# Abstract

This project consists of the development and validation of a numerical model to simulate transient responses of the ALBA's synchrotron cooling system. In particular, the work aims at studying the pumping system start-up and stop in order to detect possible problems that can lead to piping failures.

The project focus on the hydrodynamic response of the cooling system, which is part of the activities integrated in a stability and reliability plan promoted by CELLS (Consortium for the Exploitation of the Synchrotron Light Laboratory).

Flowmaster® is the 1D thermo-fluid simulation software that has been used to model the cooling system to detect dangerous pressure peaks and flow oscillations when operation conditions of the pumping stations are suddenly changed.

The first part of this project has been involved in learning and familiarizing with Flowmaster® program in order to perform correctly the simulations. Simple models have been designed to understand and learn the properties and the response influence of the components and model set-up.

The second part has involved the simulations of the actual cooling system. A model available from preliminary studies has been modified to take into account compressibility effects by replacing and adding the adequate components. In addition, it has also been necessary to create scripts and to introduce and make changes in the PID controllers in order to simulate the real ALBA synchrotron pumping system startup/stop procedures.

The normal start-up maneuver of the pumping system has been simulated and the fluid dynamic response has been analyzed. The results indicate the generation of significant pressure rises. To mitigate them, changes to the PID controller parameters have been proposed that improve the transient behavior reducing such peaks.

The simulation and analysis of pumps' shutdowns due to unexpected failures has served to identify the consequences on the system behavior and to prevent possible life-reduction conditions. The calculations have been carried out without and with simultaneous thermal regulation. For example, the results indicate that when the thermal regulation is on the consequences of the simultaneous shut-down of all pumps are mitigated.

Finally, the effect of air in the pipes has been analyzed during a pump shut-down and it has been confirmed that the transient pressure fluctuations predicted in the system are modified.





# Summary

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# A. Glossari

- BL: Beam Lines o línies experimentals.
- BO: Booster Ring o accelerador circular.
- CELLS: Consortium for the Exploitation of the Synchrotron Light Laboratory.
- CFD: Computational Fluid Dynamics
- D02: Accumulator.
- D03: Pneumatex
- EA: Experimental Area
- EX07: Heat exchanger
- PID: Proportional-integrative-derivative controller
- P07: Experimental Area pumping system
- P08: Storage Ring pumping system
- P09: Booster Ring pumping system.
- P10: Service Area pumping system.
- P11: Pumping system that bring the heated water to the heat exchangers.
- PEM: It is the Budget of Material Execution. In Catalan, stand for "*Pressupost d'Execució Material*".
- PEC: It is the Contracted Operation Budget. In Catalan, stands for "*Pressupost d'Execució per contracte*".
- SA: Service Area
- SR: Storage Ring.
- $V3V_{D02}$ : Three-way valve connected to the accumulator.
- $V3V_{P07}$ : Three way valves of the pumping ring P07
- $V3V_{\text{P08}}$ : Three way valves of the pumping ring P08
- $V3V_{P09}$ : Three way valves of the pumping ring P09
- $V3V_{P10}$ : Three way valves of the pumping ring P10





# B. ALBA beam lines

The beam lines are made of steel and once the light arrives in the line, the wavelength must be selected in order to proceed with the testing and experimentation. The beam lines that are operating in ALBA synchrotron are explained below.

- BL04 MSPD: The Materials Science and Powder Diffraction Beamline is devoted to high-resolution powder diffraction and high pressure powder diffraction using diamond anvil cells.
- BL09 Mistral soft x-ray microscopy: The full-field Transmission X-ray Microscopy beamline MISTRAL is devoted to cryo Nano-tomography in the water window and for biological applications.
- BL11 NCD non-crystalline diffraction: The experiments provide structural and dynamic information of large molecular assemblies like polymers, colloids, proteins and fibers.
- BL13 XALOC macromolecular crystallography: It aims to provide the present and future structural biology groups with a flexible and reliable tool to help in finding solutions for structures of macromolecules and complexes.
- BL22 CLÆSS core level absorption & emission spectroscopies: The beamline will provide a simultaneous and unified access to two complementary techniques; absorption and emission spectroscopes.
- BL24-CIRCE photoemission spectroscopy and microscopy: The beamline is dedicated to advanced photoemission experiments.
- BL29 BOREAS resonant absorption and scattering: It is a soft X-ray beamline dedicated to polarization spectroscopic investigations of advanced materials, as well as, applied interest.

CELLS is broaden the number of beam lines with two more, which are in construction.



# C. Technical data sheet

# C.1. Pumps data sheet

#### C.1.1. P07 (EA)



Figure C.1 P07 pump technical description



## C.1.2. P08 (SR)



Figure C.2 P08 pump technical description



## C.1.3. P09 (BO)



Figure C.3 P09 pump technical description



# C.1.4. P10 (SA)



Figure C.4 P10 pump technical description



# C.1.5. P11 (common line)



Figure C.5 P11 pump technical description



# C.1.6. Maximum torque calculations

|                                 | P07      | P08      | P09      | P10      | P11      |
|---------------------------------|----------|----------|----------|----------|----------|
| Max Power (W)                   | 56550    | 97450    | 41800    | 66150    | 59050    |
| Max Rotational<br>Speed (rad/s) | 314.16   | 314.16   | 314.16   | 314.16   | 157.08   |
| Max Torque (N⋅m)                | 180.0038 | 310.1923 | 133.0532 | 210.5615 | 375.9231 |

The formula to calculate the maximum torque is the following:

$$\Gamma \max = \frac{Pmax}{\omega max}$$

(Eq C.1)



# D. Pipe data

Last column, water speed (a) has been calculated according the formula presented in the chapter 6.2 of this project, which is the following:

$$a = \sqrt{\frac{1}{\rho(\frac{1}{k} + \frac{d}{tE})}}$$

(Eq. D.1)

Table D.1 Geometrical information of the pipes

| nominal  | OUTSIDE  | WALL      | Inner    | Inner    | speed of  |
|----------|----------|-----------|----------|----------|-----------|
|          | DIAMETRE | THICKNESS | diametre | diametre | sound in  |
| SIZE/DIN | (mm)     | (mm)      | (m)      | (mm)     | water (a) |
| 0.125/6  | 10,287   | 1,2446    | 0,008    | 7,7978   | 1434      |
| 0.25/8   | 13,716   | 1,651     | 0,010    | 10,414   | 1434      |
| 0.375/10 | 17,145   | 1,651     | 0,014    | 13,843   | 1418      |
| 0.5/15   | 21,336   | 2,1082    | 0,017    | 17,1196  | 1420      |
| 0.75/20  | 26,67    | 2,1082    | 0,022    | 22,4536  | 1402      |
| 1/25     | 33,401   | 2,7686    | 0,028    | 27,8638  | 1406      |
| 1.25/32  | 42,164   | 2,7686    | 0,037    | 36,6268  | 1384      |
| 1.5/40   | 48,26    | 2,7686    | 0,043    | 42,7228  | 1369      |
| 2/50     | 60,325   | 2,7686    | 0,055    | 54,7878  | 1341      |
| 2.5/65   | 73,025   | 3,048     | 0,067    | 66,929   | 1328      |
| 3/80     | 88,9     | 3,048     | 0,083    | 82,804   | 1297      |
| 3.5/90   | 101,5999 | 3,048     | 0,096    | 95,5039  | 1274      |
| 4/100    | 114,2999 | 3,048     | 0,108    | 108,2039 | 1253      |
| 5/125    | 141,3001 | 3,4036    | 0,134    | 134,4929 | 1233      |
| 6/150    | 168,2749 | 3,4036    | 0,161    | 161,4677 | 1196      |
| 8/200    | 219,0749 | 3,7592    | 0,212    | 211,5565 | 1159      |
| 10/250   | 273,0498 | 4,191     | 0,265    | 264,6678 | 1132      |
| 12/300   | 323,8498 | 4,572     | 0,315    | 314,7058 | 1111      |
| 14/350   | 355,5998 | 6,35      | 0,343    | 342,8998 | 1168      |
| 16/400   | 406,3998 | 6,35      | 0,394    | 393,6998 | 1136      |
| 18/450   | 457,1998 | 6,35      | 0,444    | 444,4998 | 1107      |
| 20/500   | 507,9997 | 6,35      | 0,495    | 495,2997 | 1080      |
| 22/550   | 558,7997 | 6,35      | 0,546    | 546,0997 | 1055      |
| 24/600   | 609,5997 | 6,35      | 0,597    | 596,8997 | 1032      |
| 26/650   | 660,3996 | 7,9248    | 0,645    | 644,55   | 1070      |
| 28/700   | 711,1996 | 7,9248    | 0,695    | 695,35   | 1050      |
| 30/750   | 761,9996 | 7,9248    | 0,746    | 746,15   | 1031      |
| 32/800   | 812,7996 | 7,9248    | 0,797    | 796,95   | 1013      |
| 34/850   | 863,5995 | 7,9248    | 0,848    | 847,7499 | 997       |



# E. Verification of Flowmaster software

The system that will be studied is composed by two reservoirs, an elastic pipe, a valve controller and a valve that closes very fast. The simulation will consist of a valve closing its gates very fast, within 0,01 second.



Figure E.1 Model used to verify Flowmaster software

In order to check if the closure of valve generates water hammer, it is check the pressure variation in 2<sup>nd</sup> node. It is the following:



Figure E.2 Water hammer phenomenon

Water hammer can be seen. A peak pressure of  $1,073 \cdot 10^6$  Pa is generated and the pressure reaches  $2,15 \cdot 10^6$  Pa before the wave pressure goes back and counteract the pressure generated by itself.



The calculation used by Joukowsky equation is the following:

ho = 1000 kg/m<sup>3</sup> a = 1000 m/s  $\Delta v$  = 1,075 m/s

 $\Delta p = 1000 \text{ x } 1000 \text{ x } 1,075 = 1,075 \cdot 10^{6} \text{ Pa}$ 

The theorical result and by simulation are almost the same. It is this way because the valve is closed in 0,01 seconds. As the closure is slower, the calculated values would be more different as in Joukowsky calculation, the closing is instantaneously.

Flomaster software is verified to provide plausible results.



# F. Initial simulation model

# F.1. P07



Figure F.1 Pumping P07 ring



F.2. P08



Figure F.2 Pumping P08 ring









F.4. P10







# F.5. P11



Figure F.5 Pumping P11 ring



# G. Tilting disc check valve



Other flange connections are available on request. Face to face dimensions are according to EN 558-1 (Serie 16) The RM check valve is according to PED 97/23/CE Directive Categorie I Module A Fluid Group 2(b). Under request are also available Categorie II/III.

DIR 94/9/CE (ATEX) Please contact Orbinox for information and availability of categories and zones. All valves are tested prior to shipping in accordance with the standard developed by the Quality Control Department at ORBINOX.



| STANDARD PARTS LIST |                  |  |  |
|---------------------|------------------|--|--|
| Part:               | Stainless steel: |  |  |
| 1- Body             | CF8M             |  |  |
| 2- Disc             | CF8M             |  |  |
| 3- Shaft            | AJSI 316         |  |  |
| 4- Cover            | AISI 316         |  |  |

| R Z             | Reserves the right to change spectRoations without notice<br><b>ORBINOX S.A.</b> Pol. Ind. s/n-20270 ANOETA (Spein) Tel.: +34 943 698030 - Fax: +34 943 65 | OBX 09/09 8th EDITION<br>RM-1<br>3066 e-mail:orbinox@orbinox.com |
|-----------------|--|--|
| Alter and Alter | OBEINOX CAMADA, ORBINOX USA, ORSINOX COMERCIAL, ORBINOX UK, ORBINOX FRANCE, ORBINOX GERMANT, ORBINO  | X INDIA, OBSINGX CHINA, OBSINGX S.E.A.                           |
| B (27)          | www.orbinox.com  |  |

Figure G.1 Technical description of the non-return valves





#### **DESIGN FEATURES**

## BODY:

Wafer style cast **monoblock.** Cast-in lift eye starting from DN 200 for easier installation.

#### DISC:

Cast circular, of lightweight designed for maximum strength and low inertia shape.

#### SHAFT:

Stainless steel heavy stub shafts are positioned to give maximum strength and accurate guiding to the tilting disc. One-piece design for diameters up to DN 200, two piece for diameters over DN 250.

Seal welded covers on the exterior of the valve body.



#### **OTHER OPTIONS**



# Accelerates the speed of closing.

OTHER MATERIALS OF CONSTRUCTION Special alloys such as AISI 317, 254SMO, Hastelloys, Titanium

#### FABRICATED VALVES:

**AUXILIARY SPRING** 

**ORBINOX** is equipped for in house fabrication of special valves. Depending on the design, diameter, pressures, material of construction,...





#### COUNTERWEIGHT WITH OR WITHOUT HYDRAULIC DAMPER

It is normally used in pumping stations to reduce the water harmer effect. Application of these systems requires previous study of the installation characteristics. In theses cases contact technical department is recommended.

#### SEAT TYPE



#### METAL / METAL (standard)

The sealing is effected by precision machined seats in the body and the disc edge.

| Reserves the right to change specifications without notice  | OBX 09/09 Sth EDITION     |
|---|---------------------------|
| ORBINOX S.A. Pol. Ind. s/n-20270 ANOETA (Spath) Tel.: +34 943 698030 - Fax: +34 943 653066 e-mati:orbinox@orbinox.com | RM-2                      |
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| www.orbinoz.com   |                           |

Figure G.2 Technical description of the non-return valves





| DN  | A   | В     | Ø (PN10/16/25/40-ANSI150) | ØD  | E                 | Weight (kg.) |
|-----|-----|-------|---------------------------|-----|-------------------|--------------|
| 40  | 33  | 45    | 84                        | 34  | 823               | 0,8          |
| 50  | 43  | 54    | 102,5                     | 44  | 62                | 1            |
| 65  | 46  | 64    | 121,5                     | 58  | 0.75              | 2            |
| 80  | 64  | 85    | 134,5                     | 72  | 31 <del>3</del> 3 | 3            |
| 100 | 64  | 98    | 162                       | 90  | ( <del>1</del> 7) | 4,5          |
| 125 | 70  | 116,5 | 192                       | 112 | 39 <del>8</del> 3 | 6,5          |
| 150 | 76  | 136   | 219                       | 135 | 12                | 7,5          |
| 200 | 89  | 222,5 | 273                       | 180 | 155               | 15           |
| 250 | 114 | 221   | 329                       | 225 | 182,5             | 26,5         |
| 300 | 114 | 251   | 378                       | 270 | 210               | 33,5         |
| 350 | 127 | 294   | 438                       | 315 | 240               | 54           |
| 400 | 140 | 340   | 489                       | 365 | 275               | 65,5         |
| 450 | 152 | 370   | 540                       | 410 | 300               | 92           |
| 500 | 152 | 405   | 594                       | 460 | 325               | 110          |
| 600 | 178 | 497   | 696                       | 555 | 390               | 178          |
| 700 | 229 | 616   | 800                       | 650 | 460               | 245          |
| 750 | 229 | 613   | 880                       | 650 | 485               | 310          |
| 800 | 241 | 675   | 917                       | 745 | 515               | 385          |
| 900 | 241 | 750   | 1012                      | 835 | 562               | 445          |

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RM-3
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www.erbinox.com

Figure G.3 Technical description of the non-return valves



# H. Transient simulations with the adjusted model

# H.1. Start-up and effect of pumps P07, P08, P09 and P10

Analyzing the simulations, it has been verified that the start of the pumps don't generate significant perturbations in the other pumping lines. As a consequence, the following figures will only be related to the pumping rings when their pumps are started.



### H.1.1. Start-up of P07 pump

Figure H.1 Evolution during P07 pump start-up





Figure H.2 Evolution during P07 pump start-up

![](_page_25_Figure_4.jpeg)

![](_page_25_Figure_5.jpeg)

Figure H.3 Evolution during P08 pump start-up

![](_page_25_Picture_7.jpeg)

![](_page_26_Figure_1.jpeg)

Figure H.4 Evolution during P08 pump start-up

# H.1.3. Start-up of P09 pump

![](_page_26_Figure_4.jpeg)

Figure H.5 Evolution during P09 pump start-up

![](_page_26_Picture_6.jpeg)

![](_page_27_Figure_2.jpeg)

Figure H.6 Evolution during P09 pump start-up

# H.1.4. Start-up of P10 pumps

![](_page_27_Figure_5.jpeg)

Figure H.7 Evolution during P10 pump start-up

![](_page_27_Picture_7.jpeg)

![](_page_28_Figure_1.jpeg)

Figure H.8 Evolution during P10 pump start-up

![](_page_28_Picture_3.jpeg)

![](_page_28_Picture_5.jpeg)

# H.2. Shut-down of P07 pump

![](_page_29_Figure_3.jpeg)

![](_page_29_Figure_4.jpeg)

Figure H.9 Evolution during P07 pump start-up

![](_page_29_Figure_6.jpeg)

Figure H.10 Evolution during P07 pump start-up

![](_page_29_Picture_8.jpeg)

![](_page_30_Figure_2.jpeg)

# H.2.2. Perturbations in P08 pumping line

Figure H.11 Evolution during P07 pump start-up

![](_page_30_Figure_5.jpeg)

Figure H.12 Evolution during P07 pump start-up

![](_page_30_Picture_7.jpeg)

![](_page_31_Figure_2.jpeg)

## H.2.3. Perturbations in P09 pumping line

Figure H.13 Evolution during P07 pump start-up

![](_page_31_Figure_5.jpeg)

Figure H.14 Evolution during P07 pump start-up

![](_page_31_Picture_7.jpeg)

![](_page_32_Figure_2.jpeg)

# H.2.4. Perturbations in P10 pumping line

Figure H.15 Evolution during P07 pump start-up

![](_page_32_Figure_5.jpeg)

Figure H.16 Evolution during P07 pump start-up

![](_page_32_Picture_7.jpeg)

![](_page_33_Figure_2.jpeg)

### H.2.5. Perturbations in P11 pumping line

Figure H.17 Evolution during P07 pump start-up

![](_page_33_Figure_5.jpeg)

![](_page_33_Picture_6.jpeg)

# H.3. Shut-down of P08 pumps (both two pumps simultaneously)

The stop of P08 pumps produce perturbations in the other pumping lines.

# H.3.1. Perturbations in P07 pumping line

![](_page_34_Figure_5.jpeg)

Figure H.18 Evolution during P08 pumps start-up

![](_page_34_Figure_7.jpeg)

Figure H.19 Evolution during P08 pumps start-up

![](_page_34_Picture_9.jpeg)

![](_page_35_Figure_2.jpeg)

## H.3.2. Perturbations in P09 pumping line

Figure H.20 Evolution during P08 pumps start-up

![](_page_35_Figure_5.jpeg)

Figure H.21 Evolution during P08 pumps start-up

![](_page_35_Picture_7.jpeg)

![](_page_36_Figure_2.jpeg)

#### H.3.3. Perturbations in P10 pumping line

Figure H.22 Evolution during P08 pumps start-up

![](_page_36_Figure_5.jpeg)

Figure H.23 Evolution during P08 pumps start-up

![](_page_36_Picture_7.jpeg)

# H.4. Shut-down of P09 pump

![](_page_37_Figure_3.jpeg)

#### H.4.1. Perturbations in P09 pumping line

Figure H.24 Evolution during P09 pump start-up

![](_page_37_Figure_6.jpeg)

Figure H.25 Evolution during P09 pump start-up

![](_page_37_Picture_8.jpeg)

![](_page_38_Figure_2.jpeg)

# H.4.2. Perturbations in P07 pumping line

Figure H.26 Evolution during P09 pump start-up

![](_page_38_Figure_5.jpeg)

Figure H.27 Evolution during P09 pump start-up

![](_page_38_Picture_7.jpeg)

![](_page_39_Figure_2.jpeg)

# H.4.3. Perturbations in P08 pumping line

Figure H.28 Evolution during P09 pump start-up

![](_page_39_Figure_5.jpeg)

Figure H.29 Evolution during P09 pump start-up

![](_page_39_Picture_7.jpeg)

![](_page_40_Figure_2.jpeg)

# H.4.4. Perturbations in P10 pumping line

Figure H.30 Evolution during P09 pump start-up

![](_page_40_Figure_5.jpeg)

Figure H.31 Evolution during P09 pump start-up

![](_page_40_Picture_7.jpeg)

![](_page_41_Figure_2.jpeg)

## H.4.5. Perturbations in P11 pumping line

Figure H.32 Evolution during P09 pump start-up

![](_page_41_Figure_5.jpeg)

Figure H.33 Evolution during P09 pump start-up

![](_page_41_Picture_7.jpeg)

# H.5. Shut-down of P10 pump (both two pumps simultaneously)

![](_page_42_Figure_3.jpeg)

#### H.5.1. Perturbations in P10 pumping line

Figure H.34 Evolution during P10 pumps start-up

![](_page_42_Figure_6.jpeg)

Figure H.35 Evolution during P10 pumps start-up

![](_page_42_Picture_8.jpeg)

![](_page_43_Figure_2.jpeg)

H.5.2. Perturbations in P07 pumping line

Figure H.36 Evolution during P10 pumps start-up

![](_page_43_Figure_5.jpeg)

Figure H.37 Evolution during P10 pumps start-up

![](_page_43_Picture_7.jpeg)

![](_page_44_Figure_2.jpeg)

# H.5.3. Perturbations in P08 pumping line

Figure H.38 Evolution during P10 pumps start-up

![](_page_44_Figure_5.jpeg)

Figure H.39 Evolution during P10 pumps start-up

![](_page_44_Picture_7.jpeg)

![](_page_45_Figure_2.jpeg)

## H.5.4. Perturbations in P09 pumping line

Figure H.40 Evolution during P10 pumps start-up

![](_page_45_Figure_5.jpeg)

Figure H.41 Evolution during P10 pumps start-up

![](_page_45_Picture_7.jpeg)

![](_page_46_Figure_2.jpeg)

# H.5.5. Perturbations in P11 pumping line

Figure H.42 Evolution during P10 pumps start-up

![](_page_46_Figure_5.jpeg)

Figure H.43 Evolution during P10 pumps start-up

![](_page_46_Picture_7.jpeg)

# I. Construction drawing

The following construction drawings has been helpful to adjust the Flowmaster model.

![](_page_47_Picture_4.jpeg)

![](_page_48_Figure_0.jpeg)

![](_page_49_Figure_0.jpeg)