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#### University of Aston in Birmingham

# "The Influence of Additivies on the Emissivity and Temperature of Methane Flames"

Thesis submitted for the Degree of

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by

Anthony John Myers, B.Sc.

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#### Abstract

A small, premixed methane-air flame on a cylindrical burner has been studied with special reference to the influence of additives on the radiating properties of the system. Data on changes, due to the additives, in emissivity, temperature, radiance and smoke point have been obtained. The relative effects of the additives have been assessed and some correlation between emissivity and smoke point has been demonstrated. The industrial use of the additives as a means of achieving a highly radiating flame has been shown unlikely.

#### Nomenclature

- A area of image of field stop
- A' area of near source
- A'' area of distant source

in preliminary experiment to show radiation received by pyrometer is independent of distance from source

- $A_n$  absorptivity of a single particle
- C specific heat at constant pressure
- c mass concentration of soot particles
- c, constant of Planck's Law
- constant of Planck's Law
- D diameter of flame
- d diameter of burner
- d<sub>p</sub> diameter of particle
- E<sub>n</sub> emissivity of a single particle
- f focal length of lens
- h height of inner cone of flame
- h' convection heat transfer coefficient
- I radiation incident on a typical element of a cloud of particles
- I radiation incident on a cloud of particles
- IL radiation transmitted through a cloud of particles of thickness L.
- k thermal conductivity
- k' constant in relationship between absorption coefficient and wavelength postulated by Hottel and Broughton (29)
- L thickness of a cloud of particles
- LN luminometer number as defined by Wurster et al (127)
- m constant correcting for the field of view, thermopile, etc., of the pyrometer
- N number of observations in preliminary experiment to estimate accuracy of results.
- N' number of observations in preliminary experiments to estimate significance of results.

- N<sub>H</sub> number of observations in preliminary experiments on heating of additives supply and mixing chamber
- N<sub>1</sub> number of observations in preliminary experiments on efficiency of mixing chamber.
- Nu Nusselt number
- N number of particles
- n constant correcting for the field of view, thermopile, etc. of the pyrometer
- p constant correcting for calibration distance of pyrometer
- Qr volume rate of air and methane to idealised flame
- q constant correcting for near flame distance to pyrometer
- R<sub>pyro</sub> internal resistance of pyrometer
- R radiation received by pyrometer in Schmidt method
- Re Reynolds number
- r constant correcting for distant flame distance to pyrometer
- s standard deviation of results in preliminary experiments to estimate accuracy of results
- s<sub>H</sub> standard deviation of results in preliminary experiments on heating of additive supply and mixing chamber
- standard deviation of results in preliminary experiments on efficiency of mixing chamber
- T true temperature of flame
- T<sub>B</sub> temperature of a black body
- T<sub>P</sub> brightness temperature measured at calibration distance, for estimating p, q and r.
- T<sub>Q</sub> brightness temperature for near flame for estimating p, q, and r
- TR brightness temperature for distant flame for estimating p, q and r.
- $\mathbf{T}_{\mathbf{b}}$  brightness temperature of a source

- Tw temperature of furnace walls
- T<sub>1</sub> brightness temperature of single flame
- T<sub>2</sub> brightness temperature of two flames in line with axis of pyrometer

U rad overall radiation heat transfer coefficient

 $V_{\rm p}$  pyrometer output for flame at calibration distance

 $V_{O}$  pyrometer output for near flame

V<sub>R</sub> pyrometer output for distant flame

V flame velocity

V<sub>1</sub> pyrometer output for single flame

V<sub>2</sub> pyrometer output for two flames in line with axis of pyrometer

W radiance

 $W_{B,A}$  monochromatic radiance of a blackbody

x values of emissivity, temperature or radiance in preliminary experiments to estimate accuracy of results.

x<sub>H</sub> values of emissivity, temperature or radiance in preliminary experiments on heating of additive supply and mixing chamber.

values of single flame readings in preliminary experiments on efficiency of mixing chamber.

∠ absorptivity

F,F absorptivity of radiation by a flame at temperature F from a source at temperature F.

∠ F,S absorptivity of radiation by a flame at temperature F from a source at temperature S.

correction to combined emissivities of carbon dioxide and water vapour

€ emissivity of flame

€ F emissivity of a flame at temperature F.

 $\epsilon_{\rm g}$  emissivity of gaseous components of flame

€ \_ emissivity of cloud of soot particles

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#### 1. Introduction

The discovery of natural gas, containing a high proportion of methane, in the North Sea, has led to much interest in possible uses for this fuel in Britain. The costs of locating and landing the gas are such that it is a serious competitor to oil imported from the Middle East and elsewhere.

For many furnace heating applications, oil has an inherent advantage over natural gas in that it burns with a strongly luminous flame with high emissivities. This means that high heat transfer rates by radiation from the flame can be achieved. Natural gas, when burned under conditions suitable for complete combustion, has a non-luminous flame of low emissivity and consequently there are difficulties in realising the full heating potential of such a flame.

A method of increasing the emissivity is by causing the formation of soot within the flame. Sooting may be induced by the use of additives in relatively small quantities to influence the chemical equilibria within the flame.

There has been a great deal of work on the effects of additives on various properties of flames but little on their influence on those properties relating to radiation from the flames.

The concern of this work is with the effects of additives on the radiating properties of methane flames.

The work was carried out on a small scale and consisted of a vertical premixed methane-air flame, with provision for the addition of other compounds, burning on a circular nozzle. The flame was enclosed by a water cooled calorimeter and there was provision for measuring some of the properties of the flame.

#### 2. Industrial applications of natural gas

#### 2.1 The advantages of natural gas

There have been several reviews of the possible applications of natural gas as an industrial fuel (1, 2, 3, 4). It has the following advantages compared to oil or solid fuels.

- a) The gas does not require storage facilities, pumps, preheaters, steam or special auxiliary equipment to maintain supply.
- b) Since the auxiliary equipment is eliminated the flow is even and easily controlled.
- c) The quality of the fuel is virtually constant.
- d) Natural gas is the cleanest fuel available with little carry over to choke checkers and flues. Experience has shown that on the rebuild of a typical industrial furnace up to half of the original checker bricks may be used again.
- e) Sulphur content is very low, approximately 0.2 ppm, advantageous in applications such as steel making, and presents no pollution problems. The latter point is of increasing significance.
- f) The flame is less oxidising than an oil flame. This is of particular importance in the steel industry, a reduction in the oxidation of exposed scrap during the melt period would mean a saving in the amount of pigiron necessary (5).
- g) In some cases a lower cost per therm.

Table 1

# The Performance of different types of Flames

# as calculated by Thring

¥ 245	Fuel Input Rate	Total Melt Time (hr)	Fuel req'd. ton/100 ton charge
Flame 1	4000	5.02	9.0
	2000	8.00	7.15
	1000	17.7	7.9
Flame 2	4000	3.15	5.6
	2000	6.73	6.0
	1000	17.3	7.7
Flame 3	4000	7.88	14.1
	2000	11.8	10.5
	1000	24.4	10.9
Flame 4	4000	3.76	6.7
	2000	7.54	6.75
	1000	18.5	8.25

#### 2.2 The iron and steel industry

The iron and steel industry can be expected to make extensive use of natural gas. In Russia the alternative fuels are usually coke-oven gas or blast furnace gas, natural gas has compared well with the help of incomplete combustion in the ports or in some cases with oxygen enrichment (6, 7, 8). America has tended to favour enrichment with oil for use in open-hearths, this is more efficient than oil alone but the additional expense of two fuel systems and difficulties with control are disadvantages.

Finlayson and Durrant (2) foresaw considerable scope for utilising natural gas in the metal-finishing processes and in the production of inert atmospheres. The most desirable application however would be in the open-hearth furnaces which spotlight the main disadvantage of natural gas flames. They have very low emissivities and consequently are able to radiate only a small proportion of their calorific value. The experiments of Ihospied and Reveillard (9) and Jaegle and Malmezat (10) have shown maximum emissivities of about 0.25 for thick industrial-sized flames on conventional burners.

It appears that the first documented attempt to utilise the radiation from a luminous flame was made in a glass furnace by Siemens in 1884 (11). Work on the theoretical aspects of radiation heat transfer soon followed, the major interest being in pulverised coal flames (12, 13, 14).

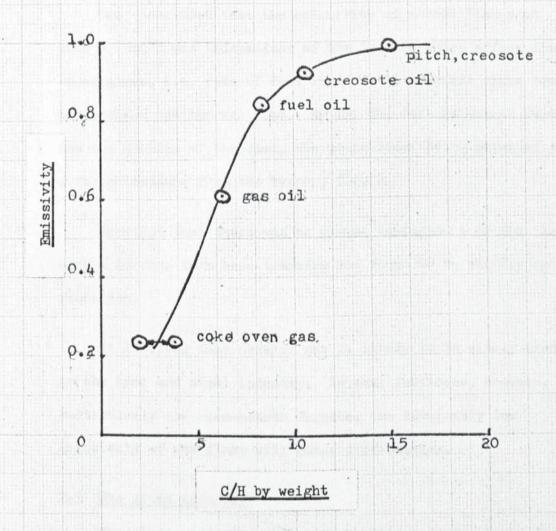
The type of flame which would be most efficient in an openhearth has been considered in some detail by Thring (15) and the results of his calculations are shown in Table 1. In flame 1 the emissivity was unity over the first sixth of the furnace, fell to 0.1 over the next sixth and was sustained at 0.1 for the remainder. This corresponds to an oil flame with rapid combustion and short flame length. The second flame was similar in that combustion was assumed to be completed in the first third but the emissivity remained at unity throughout. As yet it has not been possible to achieve a flame of this type. In flame 3 combustion occurs uniformly along the entire length of the furnace but the emissivity is as for flame 1. A coke-oven gas flame or natural gas flame with low jet momentum would burn in this manner. Finally, the fourth flame, similar to an oil or pitch-creosote flame with low jet momentum, has combustion uniform along the length and a constant emissivity of unity. A furnace charge of 100 ton was considered and a thorough examination of the heat transfer made.

It can be seen that the flames with an overall emissivity of unity give much quicker heating and are more efficient than those with lower emissivities. Furthermore, it is preferable to have a fuel which is completely burned over the first third of the furnace rather than have uniform combustion along the length.

Pearson et al (16) tested similar theories in an industrial installation and showed that radiation accounts for virtually the whole of the heat transfer within the furnace.

The heat transfer from oil and coke-oven gas flames has been studied experimentally at Ijmuiden in Holland (21). Thring and Hubbard found that for fuels with a high carbon/hydrogen





The dependence of the emissivity of industrial flames on the C/H ratio of the fuel, (Thring and Hubbard)

ratio a peak emissivity of approximately 0.8 was achieved at about half the flame length, after which it fell linearly to the non-luminous value of about 0.3. For coke-oven gas, however, the emissivity was more or less constant at 0.3 throughout the flame length.

The concluded that the emissivity of normal flames at  $1000 - 1800^{\circ}$ C was independent of the factors which affect flame temperature, i.e. rate of fuel input for a constant flame length, air preheat and thermal load. By far the most important factor was the quality of the fuel, the graph shown being obtained for a fixed distance from the burner, fig. 1.

Provided that there was no direct impingement of the flame onto a surface then heat transfer was found to be chiefly by radiation.

It is clear that natural gas is likely to be widely used in the iron and steel industry. In some instances, however, particularly the open-hearth furnace, the inherently low emissivity of its flame will cause difficulties.

#### 2.3 The glass industry

The glass industry will also provide a large demand for natural gas, its cleanliness and ease of control making it especially attractive (4). There is considerable controversy over the importance of a radiant flame in glass tanks. Most evidence (17, 18) suggests a luminous flame is required but Datschefski has achieved success with a largely non-luminous flame (19). Datschefski claimed that with a luminous flame most of the heat reradiated and reflected from the furnace roof is absorbed by the flame and not transmitted to the bath. It

should be remembered that this heat energy is not lost from the furnace and can be reradiated again to roof and bath.

In the case of a non-luminous flame, some of the radiation reaching the roof in the bands of the spectrum corresponding to carbon dioxide and water vapour is

- a) reflected at the same wavelengths to be abosrbed by the flame; and some is
- b) reradiated at all wavelengths to pass through the flame and reach the bath.

It will be realised, however, that the sum of these cannot exceed the heat originally radiated by the flame to the roof and that this has been severely limited by the non-luminous characteristics.

To achieve greater radiation from the roof it would be necessary to heat it by convection. This may have occurred in the tests of Datschefski which lasted only two months so that any damage to the refractories may not have been apparent.

Alternatively, the flame may have been close to the glass surface with convection directly to it, so that radiation from the flame and roof were less significant. The end result might well have been the overall decrease in thermal efficiency of 4% by the use of luminous flames; but due to the characteristics of the heat transfer within the glass the throughput of the furnace would have been reduced. Such data was not published.

In a later paper (48), Datschefski showed a numerical example of a mathematical model of a cross-fired regenerative

glass furnace. The results showed little difference in radiation into the bath when considering flames of different emissivities. Since the roof, flame, glass surface and exit gas temperatures were all chosen arbitrarily a more rigorous examination would be essential before drawing any definite conclusions.

For some types of glass tank at least, it seems that a luminous flame is necessary.

#### 2.4 Other industries

There are many other fields in which natural gas will find applications. Probably the largest of these is steam raising plant. In many cases the gas will be quite suitable in its natural form; many types of boilers rely on heat transfer by convection but particularly in water-tube boilers there may again be a need for a radiant flame.

Finally it will be desirable to have the facilities to use natural gas for rotary cement kilns, lime kilns, chemical processing furnaces, rotary brass melting furnaces and billet heating. Luminous flames would certainly be necessary in some of those cases.

The low emissivity of natural gas flames causes another problem in addition to that of efficiently transferring heat. The transparent nature of the flame makes it difficult to direct with accuracy, the roof of the first furnace converted to natural gas at La Consolidada steel works was burned down after only nine heats (20). Gas had tended to float upward or be

deflected off scrap and onto the roof. Increasing the emissivity of the flame would render it more easily visible and therefore simpler to direct accurately.

The industrial demand for natural gas will be considerable, therefore. In order to achieve its full potential as possibly the most important new industrial fuel available, it will be necessary to compensate for the inherently low emissivity of its flame.

#### 2.5 Heat transfer in furnaces

The main aims of furnace design are to use as small a combustion chamber volume and as small a heating surface area as possible, combined with the most efficient heat transfer which can be achieved. The thermal efficiency of the system may be rather simply defined by

efficiency = useful heat supplied to the charge calorific value of fuel consumed

High efficiencies will be favoured by good insulation of the structure, thereby reducing heat lost through the walls, and by ensuring that as little heat as possible is removed with the products of combustion. The latter requirement means that the temperature of the products of combustion must be as close as possible to the temperature of the charge; this demands high heat transfer coefficients between flame and absorbing surface to compensate for the low driving force.

In many installations, particularly those at high temperatures, significant wall losses are inevitable since extensive insulation

and the consequent high refractory temperatures would lead to rapid deterioration of the structure. In such cases the losses may be maintained as a small proportion of the calorific input by the use of high throughputs. High heat transfer coefficients are necessary to achieve this aim.

#### 2.5.1 Convection heat transfer coefficients

Considering a flame as a fluid flowing as if in a tube,

$$Nu = 0.02 \cdot Re^{0.8}$$

i.e. 
$$h = 0.02.D^{0.2} \cdot \frac{k}{2^{0.8}}$$
  $\sqrt[7]{0.8}$ 

If flame has a diameter of 2m and the thermal conductivity and viscosity are similar to those of air, then at 1000°C

$$h = 0.0023. \quad v_f \frac{\text{kcal}}{\text{m}^2.\text{h.deg K}}$$

The highest flame velocities which have been achieved are of the order of 32 m.s<sup>-1</sup> at the heating surfaces of boilers or furnaces (22). Using this value

$$h = 25 \text{ kcal.m}^{-2} \cdot h^{-1} \cdot (\text{degK})^{-1}$$

This is approximately the highest value obtainable in normal combustion chambers by forced convection.

#### 2.5.2. Radiation heat transfer coefficients

The overall radiative heat transfer coefficient is given by

$$v_{\text{rad}} = \frac{\text{Sign}(T^4 - T_w^4)}{T - T_w}$$

With a flame temperature of  $1000^{\circ}$ C, if  $T_{w} = 100^{\circ}$ C,  $U_{rad} = 3550$ .  $kcal.m.^{-2} h.^{-1} (deg K)^{-1}$ if  $T_{w} = 900^{\circ}$ C,  $U_{rad} = 1420$ .  $kcal.m.^{-2} h.^{-1} (deg K)^{-1}$ 

Hence it can be seen that radiation is markedly more significant than convection provided that the emissivity is greater than about 0.1. Therefore the most important method of improving the heat transfer is by increasing the emissivity of the flame.

#### The radiation from a flame

#### 3.1 Factors influencing the radiation from a flame

The radiation from a flame is a function of its temperature, the types of molecule present, the shape and thickness of the flame and the concentration, size and nature of any solid particles which may be present.

The temperature influences the amounts of radiation from the several different sources in the flame. The most significant sources of non-luminous radiation are carbon dioxide and water vapour present in the flames, these molecules and to a lesser extent carbon monoxide molecules radiate in small discrete bands of wavelength outside the visible spectrum. Their contribution can be determined from graphs published by Hottel (23) and others (24, 25).

The recombination of free atoms and radicals may cause a small amount of radiation as experienced by Saleh (26). More significant may be chemiluminescence which has been measured at up to 5% of the radiation of a premixed flame (27, 28).

Solid particles most frequently found in flames are soot,

Hottel and Broughton (29) have shown that the monochromatic

emissivity of a cloud of soot particles was given by

$$\epsilon_{s\lambda} = 1 - \exp(-c.L.f(\lambda))$$

f(A) represents the absorption coefficient which may be a function of particle size. For soot, after reviewing previous

experimental results of others, Hottel and Broughton suggested that in the infrared above 0.8

$$f(\lambda) = \frac{k!}{\lambda^{0.95}}$$

and in the visible spectrum

$$f(\lambda) = \frac{k^1}{\lambda^{1.39}}$$

Experimental evidence, at Ijmuiden in particular, has confirmed a relationship of this sort (21, 30).

Thring and Hubbard (21) and Sato et al (37) have obtained the following relationship between flame emissivity and the separate soot and gas emissivities of the flame

$$\in = \epsilon_s + \epsilon_g - \epsilon_s \cdot \epsilon_g$$

It is clear that the emissivities of the soot and the gases are equally weighted in determining the resultant emissivity of the flame. However, because the gases radiate over such a restricted portion of the infrared there is little scope for increasing their contribution. A better way of increasing the flame emissivity is by loading the gas with small uniformly dispersed solid particles at gas temperature, radiating at wavelengths covering a wide section of the infrared spectrum. It is very important that the flame temperature is not greatly reduced by the introduction or formation of the particles since the radiation is proportional to the fourth power of the absolute temperature.

#### 3.2 Radiation from clouds of particles in a flame

As has previously been stated for radiation, the emissivity of a flame is a function of its temperature, the types of molecule present, the shape and thickness of the flame and the size, concentration and nature of any solid particles which may be present.

The absorptivity and transmissivity are functions of the same properties as the emissivity and also of the quality of the incident radiation as defined by its spectral distribution.

Since the wavelengths of any radiation are dependent on the temperature of its source it follows that the absorptivity of particles in a flame will be a function of the temperature of the source of the radiation being absorbed. The absorptivity and transmissivity may, therefore, be given subscripts to indicate first the temperature of the flame and secondly the temperature of the source.

i.e. 
$$\ll_{\text{F.S}}$$
 and  $\succsim_{\text{F.S}}$ 

Shack has estimated that for soot particles in a flame the temperature difference between the particles and the gases is of the order of 1°C (31). The particles gain heat from the gases by convection to replace that lost to the surroundings by radiation. For most purposes, then, the particles can be considered to be in thermal equilibrium and Kirchoff's law will apply. For bodies in thermal equilibrium the emissivity at a particular temperature is equal to the absorptivity which the body exhibits for radiation from a black body at the same temperature.

i.e. 
$$\in_{\mathbb{F}} = \bowtie_{\mathbb{F},\mathbb{F}}$$

For certain bodies the absorptivity is independent of the quality of the incident radiation and is, therefore, also independent of the source of radiation. These are referred to as grey bodies and

for all flame and source temperatures. Bodies whose absorptivity is a function of the wavelength of the incident radiation are described as non-grey.

Thus there are two situations in which it can be assumed for a flame that

- a) If the background source of the radiation is at the same temperature as the flame.
- b) If the flame can be considered as a grey body.

The Schmidt method as used by the I.F.R.F. at Ijmuiden for emissivity and temperature determination assumes that emissivity and absorptivity are equal although the flame and background temperatures are different (32,33). Luminous flames were found to give satisfactory results, acting as grey bodies; Hottel suggests that this is not always the case (34) and non-luminous flames are decidely non-grey. This probably accounts for the errors encountered with non-luminous flames of coke-oven gas.

Emissivities of 0.345 have been measured when in theory they should

be about 0.21 (35). It is possible to avoid this difficulty
by the use of a black body background of variable temperature.

Series of readings for backgrounds above and below the flame temperature are taken and the desired values obtained by interpolation.

Such a system has been used but the experimental procedure and treatment of results was extremely tedious and time consuming (36).

By assuming Kirchoff's law it can be shown that for a cloud of black particles

Using the data of Thring (38) for fly ash

$$C_{\rm p} = 2.10^3 \, \rm kcal.m^{-3}$$
 $C_{\rm p} = 0.25 \, \rm kcal.kg^{-1}. \, (deg \, C)^{-1}$ 

A flame of thickness 1 m at 1500°C has

$$C_p \simeq 0.3 \text{ kcal.kg}^{-1} \cdot (\text{deg C})^{-1}$$

If the particles have no more than 10% of the heat capacity of the gases to avoid excessive cooling then

whence maximum concentration of particles 4 0.023 kg.m-1

From equation 3.1 can be obtained Table 2, showing particle cloud emissivity for different particle sizes.

Table 2

<sub>ф</sub> ( <b>h</b> )	<u>L</u>	$\frac{3}{2} \cdot \stackrel{\mathbf{c}}{\not{\rho}} \cdot \frac{\mathbf{L}}{\mathbf{d}_{\mathbf{p}}}$	٤
103	103	0.015	0.015
102	102	0.15	0.14
10	10	1.5	0.78
1	1	15	1.0

The non-luminous emissivity of such a flame can be calculated from Hottel's curves (23). At 1 atmosphere with 14% carbon dioxide and 11.5% water

$$\epsilon_{\text{CO}_2} = 0.12 \quad \epsilon_{\text{H}_2\text{O}} = 0.12 \quad \Delta \epsilon = 0.04$$

$$\epsilon_{\text{g}} = \epsilon_{\text{CO}_2} + \epsilon_{\text{H}_2\text{O}} - \Delta \epsilon$$

$$= 0.20$$

A 10% reduction in the flame temperature due to the presence of the particles means a reduction of 40% in  $T_f^{\ 4}$ . For this to be compensated and for a gain in overall radiant heat transfer it is clear that the particles must be rather less than  $10\,\mu$ .

The addition of such particles to a flame would necessitate extremely fine and therefore expensive grinding operations. They

are, however, to be found as the fly ash remaining after the combustion of a pulverised coal flame. Such particles also occur in cement kilns when the dust from the limestone-clay feedstock falls through the flame.

It has been found that soot particles formed in flames fulfil the above requirements with regard to size. This possibility has the advantage that as much of the soot is ultimately burned there is likely to be less loss in overall heat transfer due to cooling of the flame.

At Ijmuiden electron microscopic studies of soot formed in the experimental furnaces have shown the particles to be individual particles in a size range 0.01 - 0.08 M, a mean of approximately 0.04 M (38).

# 3.3 Previous attempts to produce luminous flames

As early as 1927, Haslam and Boyer (50) found that radiation accounted for a minimum of 25 - 30% of the heat transfer within a furnace. The transition from a non-luminous to a luminous flame, in their case by the use of different fuels, caused an increase in rate of heat transfer of up to four times. They suggested that some means of innoculating non-luminous flames with an illuminating agent would find many industrial applications.

#### 3.3.1 Preheating

While experimenting with glass tank burners Trinks and Keller (14) found that preheating the air gave useful increases in the luminosity of the flame. Preheating the natural gas, however,

caused a reduction. The reasons for this are not clear but the aerodynamics of the flow around the burner may well have altered with preheat temperatures up to 1100°C causing expansion of the reactants.

In 1965 Guenbant and Gaydon (53, 54) observed an increase in thermal radiation from a methane-air flame which had been preheated to 750°C. They claimed this was due to a change in the shape of the flame and not simply a function of the rate of reaction in the flame front.

Ihospied and Reveillard (9, 51) experimented with preheating of the air supply to an experimental furnace, although there was some cracking of the methane early in the flame the maximum emissivity was scarcely affected, in contrast to the results of Trinks and Keller. Jaegle and Malmezat (10) confirmed the findings of Lhospied and Reveillard.

#### 3.3.2 Precracking

Recent research in Europe has been concentrated on the production of carbon particles by thermal cracking of the methane before it enters the combustion chamber. This precracking may be achieved by heat transfer via checkers and flues with hot products of combustion leaving the furnace. By applying this heating to about one fifth of the natural gas supplied to an open hearth furnace Andoniev et al (6) produced a luminous flame containing 15-20 g soot/ft<sup>3</sup> natural gas. The performance of the hearth was similar to that using coke oven or blast furnace gas.

Using a similar technique Levitasov et al (7) increased the soot concentration of a natural gas flame from 6.9 to 14.8 g.m<sup>-3</sup>. The throughput of the furnace was increased by 10%, the flame temperature by 20-60°C and the utilisation of heat in the refining period by 15.10<sup>3</sup> kcal.m<sup>-2</sup>.h<sup>-1</sup>. Foaming in the hearth at the end of the melting and the beginning of the refining period was markedly reduced.

Considerable work is being done in France, Holland and Britain on the design of new types of burner to produce a luminous flame (9, 10, 51, 52). Most of these burners have two stages, in the first a portion of the methane is burned with a less than stoichiometric supply of air. The excess methane is cracked by the heat of combustion and the resulting carbon particles are carried forward by the primary combustion products to the second stage where they are mixed with the main methane supply. Stable flames and emissivities of about 0.8 can be achieved by these burners.

### 3.3.3 Mixed fuels

The suggestion of Haslam and Boyer of innoculating nonluminous flames with an illuminating agent has been considerably exploited in both America and Russia (55, 56, 57). In America it has been common practice to run open hearth furnaces with up to 50% of the thermal demand coming from oil injected with the natural gas.

There has been more use of mixed gas and solid fuels such

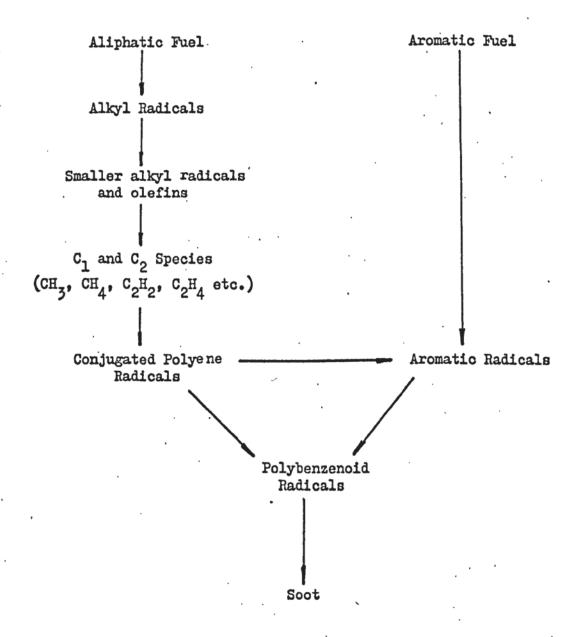
as pulverised coal in Russia. By the use of 6 - 12 g crude oil/m<sup>3</sup> peat gas Zakharikov and Rozhanskii (58) were able to shorten the melt time in a glass tank by 14 - 28% and effect fuel economics of 11 - 34%.

Kapustin et al (8) found that the injection of a little oxygen into the air supply of an open hearth furnace greatly increased the luminosity of the flame. There was a reduction in melt times and saving in fuel costs. A possible explanation is that the additional oxygen at the burner port caused extremely rapid combustion. The poorly mixed condition of fuel and oxidant at this point would cause very high local temperatures, resulting in thermal cracking of some of the methane providing carbon particles and a luminous flame.

Solid fuels in particulate form were added to a natural gas flame by Dewerth and Zalavadia (59) as 4 - 15 w/w % of the gas. Increases in radiation of up to 270% were recorded with charcoal less effective than coke but more so than pulverised coal. In addition, Dewerth and Zalavadia thought it might be possible and useful to inject some of the raw materials of certain furnaces into the flame. Lime, silica and cement were tested and the maximum increase in radiation was found to be 55%. These authors concluded that the fuels were superior due to their additional heat input into the furnace and the increase in the partial pressure of carbon dioxide. Before this conclusion could be accepted information on particle size, size distribution and surface emissivity would need to be obtained.

## Figure 2

# The Mechanism of Soot Formation according to Thomas



Dewerth and Zalavadia also showed that the addition of pulverised coal caused a greater increase in radiation than did a similar quantity of methane.

#### 3.4 The formation of soot in flames

Though it has long been realised that the luminosity of hydrocarbon flames is caused largely by the presence of particles consisting mainly of carbon, the mechanism of their formation is by no means clear (60). Furthermore the composition of these particles has been found to vary considerably (61, 62), the main constituents being carbon and hydrogen. These particles will be referred to as soot rather than carbon.

There have been many theories put forward to explain soot formation in a flame as opposed to simple thermal cracking of the fuel. These have been summarised by Gaydon and Wolfhard (60). In general all those theories agree that polymerisation occurs at some point, the difficulty being to establish at what stage in the process. Porter (63) and Booth (64) have suggested that there may be two mechanisms occurring simultaneously producing two different types of soot. Particles which have polymerised sufficiently in the preheating zone are able to survive pyrolysis while smaller particles may be broken down in the reaction zone to acetylene and hydrogen before forming soot.

In 1961 Thomas (65) suggested the mechanism shown in fig. 2.

Aliphatic fuels first form radicals by hydrogen abstraction. The resulting weak carbon skeleton then breaks easily giving simple radicals and olefins, the latter may be dehydrogenated to polyolefins

or broken down further. Once conjugated species have formed they are less likely to be broken down and are more likely to undergo addition. The conjugated polyene radicals cyclize to polybenzenoid radicals which amalgamete to form soot. Aromatic fuels also form radicals as a first step but these already have a stable conjugated structure and can build up immediately.

This theory accounts for the dark space between the reaction zone and luminous zone in an aliphatic premixed flame. The aromatic route being quicker, there is no such dark space in flames of this type.

Halogenated compounds which encourage soot formation as additives probably increase the rate of dehydrogenation, thereby assisting in olefin and polyene formation and ring closure. Sulphur trioxide might yield similar results by way of dehydrogenation.

The non-equilibrium formation of soot as detected by Street and Thomas prompted Millikan (66) to analyse large, flat, premixed ethylene-air flames. He found that about 1% acetylene and a smaller amount of methane are present in the reaction zone and postulated that pyrolysis of the acetylene is the source of soot. Millikan thought the dark space is due to oxidation of some of the acetylene by an above equilibrium concentration of OH radicals. As the OH concentration decreases so pyrolosis predominates and soot formation is observed in the luminous zone.

Ray and Long (61) also established that acetylene is an important step in the process but distinguished between "carbonaceous residue" and pentane-soluble material present in the soot. While

studying propane-air diffusion flames they found that 90% of the pentane-soluble material was polycyclic aromatics and concluded that these were not intermediates in the formation of the carbonaceous residue but byproducts.

Foster (67) has calculated that above 1140°K 95% of the carbon should be deposited during the pyrolysis of pure methane. Experimental results show a maximum yield of approximately 50% (63). The work of Tesner (68) suggests that hydrogen formed during the pyrolysis retards the production of soot. Tesner found that an increase in hydrogen concentration from 0 to 10% caused a 75% decrease in the soot surface growth rate. The pressure of oxygen means a reduction in the yield due to an increase in the formation of the oxides of carbon. From purely chemical equilibrium considerations carbon can only exist when the oxygen/carbon atom ratio is less than 1.0, i.e. less than one third of the stoichiometric air for hydrocarbons. In practice carbon has been observed at much higher ratios, deviation from equilibrium being accentuated at increased pressures up to 15 or 20 atomospheres. Francies (69) and Narasimham and Foster (70) have shown that the decomposition of hydrocarbons can occur much more rapidly than the oxidation reactions of hydrocarbons or soot by carbon dioxide or water vapour which would be necessary to prevent the formation of soot.

Considerable work has been done on soot formation by Homann and Wagner (71, 72, 73). They classified premixed flames into two types:

- 1. The acetylene type in which soot appears above the whole flame cone.
- 2. Other types showing soot formation at the tip of the cone,

either close to the flame front or a short distance above it.

Methane falls into the first category along with other light hydrocarbons.

Soot formation is not exclusively controlled by the flame chemistry. In addition to the effects of temperature and pressure the aerodynamics of the flame play an important role. Identical fuel-oxident mixtures which, by the use of different burners, exhibit a variety of shapes of flame fronts can have markedly differing tendencies to soot. It seems that the greater the curvature of the flame front so more soot formation will occur. In part this may be due to hydrogen atoms and light radicals diffusing back into the unburnt gas just upstream of the reaction zone, heating the fuel molecules and causing pyrolysis and thence yielding a flame of the second type. When this diffusion does not occur a flame of the first type is obtained.

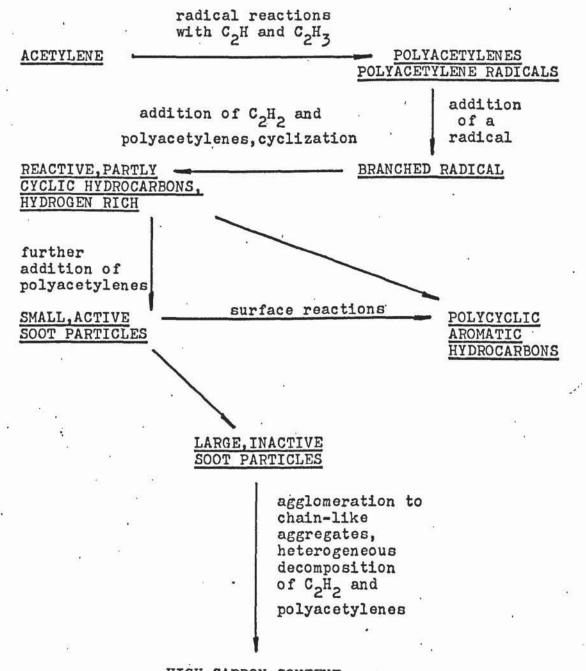
The effects of temperature and pressure differ depending upon the physical structure of the flame, e.g. in a flat benzene-air flame of the first type, soot formation is reduced by raising the temperature. If an identical mixture is burned as a cellular flame, by using a cylindrical premixing burner, however, the opposite effect is observed.

Homann and Wagner determined concentrations of CO, H<sub>2</sub>, H<sub>2</sub>O, CO<sub>2</sub>, C<sub>2</sub>H<sub>2</sub> and polyacetylenes throughout an acetylene-oxygen flame. In addition, to these molecules they found the following radicals, C<sub>2</sub>H<sub>3</sub>, C<sub>2</sub>H, C<sub>2</sub>, CH, H, O and OH together with polycyclic aromatics such as naphthalene, pyrene, phenanthracene, 3, 4-benz-pyrene and

## Figure 3

Mechanism of soot formation according to

Homann and Wagner



coronene. They concluded that in the acetylene flame most of
the soot formation takes place in the burning zone since radicals
occur there in high concentrations following oxidation processes.
The burnt gases from this zone contain soot particles, acetylene,
polyacetylenes, polycyclic aromatics, water vapour and carbon
dioxide. Further soot deposition is very slow.

The polyacetylenes are thought to be formed by way of

$$c_{2}^{H_{2}} + c_{2}^{H_{2}} + c_{2}^{H_{2}} + c_{2}^{H_{2}} + c_{2}^{H_{2}} + c_{2}^{H_{3}} + c_{4}^{H_{4}} + c_{4}^{H_{2}}$$

if there is an excess of acetylene. At the point in the flame where the concentration of polyacetylenes is at a maximum, polycyclic molecules begin to appear. It is suggested that the addition of hydrocarbon radicals to large linear hydrocarbon molecules, the polyacetylenes, leads to the formation of branched radicals. These either form closed rings with side chains, polynuclear aromatic molecules, or add further unsaturated molecules to become the precursors of soot particles having a variety of configurations. Homann and Wagner agree with Ray and Long that pure polycyclic aromatics are byproducts and not intermediates in the process of most formation. Their mechanism is summarised in fig. 3.

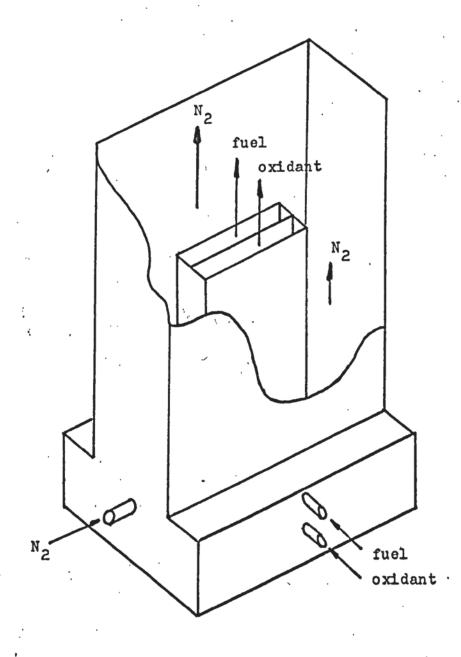
#### 4. Additives and flames

There has been a considerable amount of research into additives and their effects on flames. The reasons for using additives are diverse, possibly to achieve a particularly desired characteristic such as high or low flame speed or measuring effects of the addition as a means of establishing reaction mechanisms. An attempt has been made here to pick out the most significant and interesting points from the literature.

In 1945 Whittingham et al (74, 75, 76) obtained the spectra of a variety of premixed flames to which sulphur dioxide and trioxide were added. With coal gas burning on a bunsen burner they found that sulphur dioxide caused a reduction in the intensity of the carbon continium. Sulphur trioxide when introduced into the air stream in proportions as low as 0.1% v/v greatly increased the soot formation. They felt that the sulphur trioxide probably initiated polymerisation of hydrocarbons which were then thermally decomposed to solid carbon. The town gas used was specified as

CO<sub>2</sub> 3.5 % v/v CO 11.2 CH<sub>4</sub> 23.3 H<sub>2</sub> 46.1 O<sub>2</sub> 0.7 h/c 2.9 N<sub>2</sub> 12.3

Sulphur dioxide had the same effect on methane as on coal gas but sulphur trioxide was not tried with this gas though it



Wolfhard-Parker flat flame burner

had little effect on hydrogen or carbon monoxide.

Wolfhard and Parker (77) published further work on these additives using their flat-flame burner shown in fig. 4.

Methane-oxygen and ethane-oxygen flames were used. Sulphur dioxide addition resulted in a decrease in luminosity whether present in the fuel or oxidant stream of both flames. Sulphur trioxide in quantities of the order of those used by Whittingham et al had no significant effect on the soot formation but if present in larger quantities than sulphur dioxide then a similar reduction in luminosity occurred. Wolfhard and Parker therefore suggested that in cold regions sulphur trioxide had no promoting influence on soot formation and dissociates according to

$$S0_3 - S0_2 + \frac{1}{2}0_2$$

Furthermore, above 800°C on the fuel side where there is no oxygen and above 1500°C on the oxygen side, such dissociation is complete and no sulphur trioxide reaches the reaction zone.

The addition of nitric oxide to a methane-oxygen flame has been found to cause a slight decrease in soot formation (78).

Nitric oxide is known to be a chain-breaking molecule and so this result suggests that soot formation is not a chain reaction as has been suggested.

Extending their work to ethylene-nitric oxide flat diffusion flames, Wolfhard and Parker (79) observed marked similarities to

ethylene-oxygen flames. With nitrogen dioxide-ethylene flames, however, two reaction zones were detected, separated by up to 2 mm. In the first zone the nitrogen dioxide was thought to undergo decomposition to nitric oxide, the reverse reaction to which causes a yellow continuum to be radiated. The second zone was very similar to that in a nitric oxide-ethylene flame. In fuel rich mixtures a soot zone sometimes formed between the reaction zones and was thought to be due to thermal cracking of the fuel in the hot first zone. The two zones were observed in methane flames also.

Arthur (80) found that the addition of large quantities, up to 45% of the total volume, of carbon dioxide or nitrogen would suppress soot formation in a methane-air diffusion flame. This was claimed to be not solely due to cooling of the flame since preheating the gases did not change the critical additive concentration necessary to cause complete suppression of soot formation. Others, however, have found preheat to enhance soot formation to varying degrees (9, 10, 81).

Mellish and Linnett (82) and Clingman et al (83) studied the addition to or substitution of the nitrogen in hydrocarbon-air flames with regard to flame velocity and inflammability limits.

No changes in the luminosity of the flames were noted.

Methane-air diffusion flames on both a normal cylindrical burner and a reversed type with air passing through a tube into an atmosphere of methane were examined by Arthur and Napier (84).

#### Street and Thomas

Additive Effect

Carbon dioxide No change

Carbon monoxide Yellow streak increased in size and

intensity, cone remained unchanged.

Hydrogen Slight increase in yellow luminosity

and a considerable increase in the intensity of the green colour in the

cone.

Chlorine The cone, normally blue-green in colour,

became intensely green, the flame elongated and the yellow tip increased

in size and intensity.

Methyl bromide Flame cones became much greener and more intense, the yellow tip increased

in size, at large additive rates an orange glow appeared above the flame.

Carbon tetra- Marked increase in yellow luminosity and change of blue-green cone to intense

green, elongation of the flames.

Nitrogen peroxide Yellow tip disappeared and a greenish

white shroud surrounded the flame.

Amyl nitrate Small increase in yellow luminous tip,

an increase in air supply was necessary

to eliminate the yellow.

Tetraethyl-lead Mauve glow surrounding the cone, a

greysih black deposit quickly formed on the glass sleeve and proved to be mainly

metallic lead with traces of lead

monoxide and lead sulphide.

Sulphur dioxide Violet glow above the cone.

Sulphur trioxide Benzene flame was extinguished and and sulphuric axid copious carbonaceous deposits formed

in the burner tube; large amounts of carbon started to form in the kerosene flame which took on the appearance of

a very rich flame.

Carbon disulphide Yellow streak increased in size but remained weak in intensity, a violet glow appeared above the cone and sulphur

was deposited on the burner rim and the

glass sleeve.

# Table 3 (cont.)

Additive

Effect

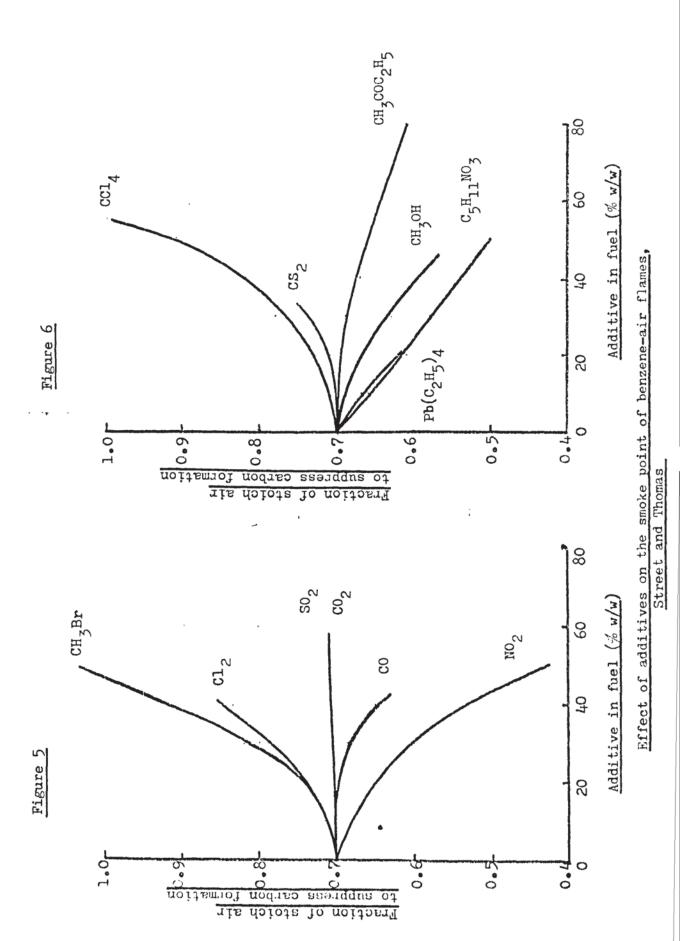
Methyl ethyl

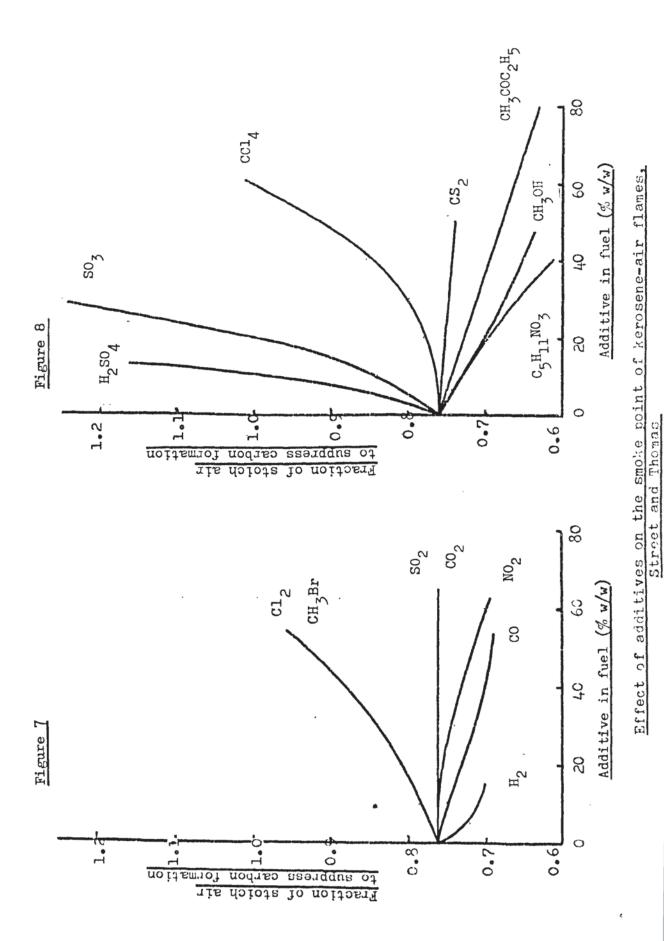
ketone

Increase in size of yellow streak for both fuels and a decrease in green component of the benzene flame.

Methyl alcohol

Yellow streak increased in size but not in intensity, an orange glow appeared over the cone and proved to be due to sodium.





The smoke points, defined by the air-fuel ratio when soot formation is just visible to the naked eye, were determined with several additives which all decreased soot formation. In order of effectiveness they were sulphur dioxide, carbon dioxide, nitrogen and carbon monoxide; up to 300% v/v of the methane were necessary for complete suppression. Light output from the flames was measured by a photocell. Much smaller additions were required to cause reductions of intensity of the order of ten times. The wavelengths of radiation to which the photocell was responsive are not known and so the results can hardly be considered as a measure of soot formation. The addition of about 0.5% sulphur trioxide had no obvious visual effects.

The major work on the influence of additives upon soot formation in flames was performed by Street and Thomas in 1955 (85). The effect of a variety of additives on the 'critical concentration of air required to suppress carbon formation in kerosene and benzene flames' was measured. A premixed flame with no secondary air was employed, the critical air flow being a mean of the air flows when yellow luminosity could just be observed continuously and when the yellow luminosity had just disappeared. Additives used and their effects were as shown in Table 3 and figs. 5 - 8.

Street and Thomas explain the increase in soot formation in the presence of halogenated compounds by their known properties as catalysts for the polymerisation of hydrocarbons. In addition Rust and Vaughan (86) have shown that alkyl halides reduce the chain splitting which occurs during low temperature oxidation of hydrocarbons. It is possible, therefore, that large molecules necessary for soot formation are preserved by the halides. It seems likely that the use of halogenated compounds to increase soot yields would be less successful with lower hydrocarbons, especially methane, in which large molecules are not present intially.

Both sulphur trioxide and sulphuric acid caused large quantities of soot to be formed in both flames, the former being very difficult to control. The additives reacted with the benzene depositing 'carbon' in the burner tube, even before the flame, this did not occur with kerosene. The reaction rates may be slower with kerosene than with benzene or the fact that the benzene was a homogeneous vapour whereas the kerosene was a mist of droplets may be the explanation.

Methyl alcohol, methyl ethyl ketone, carbon monoxide and amyl nitrate all suppressed soot formation, probably due to a simple blending process rather than affecting the normal reactions in the benzene and kerosene flames.

Nitrogen dioxide had a greater effect on the benzene flame than on the kerosene flame. It is thought that in the former case the dioxide dissociates according to

whereas in the latter by

$$4N0_2 - N_2 + 2N0$$

followed by partial dissociation of the nitric oxide. There is thus more oxygen available in the benzene flame. The benzene flame is thought to be hotter causing the more complete dissociation.

Tetra-ethyl lead suppressed soot, probably because of its known chain-breaking properties.

Water, very difficult to vaporise due to wetting of the heating surface of the vaporiser, carbon disulphide and sulphur dioxide had no effect on the critical air ratio up to 50% w/w. The sulphur dioxide reduced the intensity of the yellow luminosity but not the limits, contrary to the evidence of others (74, 75, 76, 77).

Nitrogen was found to provide soot formation and oxygen to suppress it more than in proportion to their concentrations.

Methane-chlorine trifluoride flat diffusion flames were studied by Skirrow and Wolfhard (87). The carbon continium was very strong and C<sub>2</sub> and C emitters were also detected, the chlorine trifluoride being contaminated by traces of nitrogen. The addition of oxygen or nitrogen caused marked reductions in the intensity of the continuum. Thehigh cost of chlorine trifluoride, approximately £80/200 ft.<sup>3</sup>, would probably eliminate it as a potential additive for industrial use.

While measuring temperature profiles of hydrogen-air diffusion

flames Stephens et al (88) used a sample of calcium oxide as a luminescor to locate the position of the inner cone. Addition of small quantities of oxygen led to temperatures above the calculated adiabatic flame temperature. This was thought to be due to the catalytic action of the surface of the thermocouples, bare metal couples producing higher temperatures than ceramic ones but a complete explanation cannot be provided.

Fenimore et al (89) determined the smoke point of several premixed hydrocarbon-air flames with gaseous additives. Nitrogen, argon, helium, carbon monoxide, carbon dioxide and especially water were all found to suppress soot formation. Adding 20% W/W hydrogen to propane caused an increase in the critical propane/air ratio of 13%, i.e. soot formation was encouraged, as opposed to the results of Street and Thomas for kerosene and benzene fuels.

The compositions of diffusion flames of simple alcohols burning in air were measured by Smith and Gordon (90). They found that methanol flames produced no carbon while ethanol flames exhibited a small amount of yellow tipping, n-propanol and i-propanol flames produced relatively large quantities of soot.

The same burner with a wool wick and natural circulation of air was used in all cases. It is interesting to note that the methanol flame had less than 0.004% w/w acetylene whereas the other three flames all had about 0.1% w/w benzene and 1-2% w/w acetylene. The quantity of soot formed did not, therefore, appear to depend on the concentrations of acetylene or benzene though more work would be necessary to be sure.

During the course of their work on the flame front structures of lean diborane-air flames Breisacher et al (91) added small quantities, less than 6% v/v of methane and propane. A large proportion of the hydrocarbon was cracked in the first reaction zone giving rise to a luminous region and the formation of ethylene as well as mot. At sufficiently high temperatures a second reaction zone was established downstream where the oxidation of the hydrocarbons was completed. It is unlikely that reversing the situation and adding small quantities of diborane to methane flames would produce a similar luminous zone.

As part of a study on the mechanism of soot formation Cole and Minkoff (92) made a spectroscopic analysis of methane-oxygen flames supported on a Wolfhard-Parker burner (77). Hydrogen, water, ethanol, tetralin, carbon dioxide and ethylene were added as 3% w/w of the fuel and no change in the intensity of the carbon zone was observed. It was necessary to add 15% w/w of methyl bromide to significantly increase the soot formation though it was impossible to deduce whether the carbon came from the methyl bromide or the methane. 3 - 4 % w/w of sulphur dioxide rendered the luminous zone almost invisible and 1% w/w acetylene had no measureable effect.

Tesner et al (93) differentiated between the mass of carbon and the number of particles of soot at different points in methaneair diffusion flames. The addition of nitrogen or hydrogen caused a reduction in the mass of soot formed although the number of particles increased for additive concentrations up to 20% v/v of nitrogen and 34% v/v of hydrogen after which it decreased again.

Several hydrocarbons were also added (ethane, ethylene, acetylene, propane, hexane, benzene and naphthalene). The number of particles was greatly increased by benzene and naphthalene but hardly affected by acetylene or ethylene. The mass of soot was increased by all the additives, the increase being greater for the higher molecular weights and the higher concentrations of additive. Details of the masses of soot formed might show whether the type of molecule or the molecular weight is more significant but these are not available.

Fuels containing mono- and polycyclic aromatic compounds were investigated by Schirmer et al (94) in gas turbine combustors. At pressures from 5 - 15 atmospheres high flame emissivities were noted with the formation of large carbon particles reradiating as black bodies. At these elevated pressures the polycyclic compounds produced greater increases in radiation from the flame than did the monocyclic compounds. At atmospheric pressure the aromatics had no measureable effect on the radiation though this may have been due to recirculating products of combustion screening the radiation measurement apparatus.

When a stream of nitrogen is passed through an electrical discharge at low pressures the effluent gas stream emits a yellow glow and is known as 'active' nitrogen. It is generally agreed that the reactive species present is atomic nitrogen. Jennings and Linnett examined (95) the spectra produced by introducing a variety of compounds into a stream of active nitrogen. When

studying hydrocarbons they tested flames of acetylene,
ethylene and methane and found that the brightness decreased
as the carbon to hydrogen ratio decreased. Methane produced
one of the weakest flames of all, being lilac in colour due to
the emission, in the red and violet regions of the spectrum,
of CN.

Schmidt and Schulze (96) encountered difficulties caused by soot formation in burners using natural gas. The trouble was traced to small quantities of iron pentacarbonyl which were formed during the underground storage of the gas despite its low carbon monoxide content. The concentration was reported as less than that which could be detected by ordinary analytical means. Bonne et al (97), however, found no pronounced carbon continuum when studying the spectra of methane-air flames containing small quantities of iron pentacarbonyl. As little as 0.01% w/w was sufficient to reduce the flame speed by 25%.

The reduction in propagation velocity of premixed methane-air flames by the addition of metal salts has been investigated by Rosser et al (98). 0.006 mg.cm<sup>-3</sup> of sodium carbonate was sufficient to reduce the velocity from 65 cm.s<sup>-1</sup> to 16 cm.s<sup>-1</sup>. No observations on soot formation or luminosity of the flames were made. While at first sight soot formation might be expected to occur as inhibition takes place, the theory of Rosser et al that the metal salts deactivate such species as C<sub>2</sub>, CH and OH suggests that there may be a consequent shortage of the radicals necessary for building larger molecules. The inherent reduction in flame speed renders unlikely the industrial

use of any similar additive which might induce soot formation.

The relatively low flame speed of methane alone is proving to
be a source of difficulty in many installations.

Fenimore and Jones (99) studied the inhibition of hydrogenoxygen flames by methyl bromide, probably on a premixed flame thought this is not clear. They concluded that methyl radicals were formed in the flame according to

$$H + CH_3Br = CH_3 + HBr$$

and these react with oxygen atoms causing inhibition of the flame

In addition they suggested that methane would yield methyl radicals and hydrogen in a similar manner

$$CH_A + H \longrightarrow CH_3 + H_2$$

the methyl radical again inhibiting the flame. Their study of
the flammability limits of hydrogen-methane and hydrogen-methyl
bromide mixtures support this view and so adding methyl bromide
to a methane flame is likely to have a similar effect as using
a fuel-rich mixture. The formation of formaldehyde as shown above
suggests that the addition of formaldehyde to a methane flame
might increase the rate of propagation of the flame and reduce
the likelihood of soot formation.

The effects of several known inhibitors on methane-air and ethylene-air diffusion flames have been measured by Ibiricu and

Gaydon (100). All were found to encourage soot formation when present in quantities up to 5% v/v. In order of effectiveness they were carbon tetrachloride, phosphorus oxychloride, chlorine and methyl bromide; the intensity of the carbon continuum increasing by a factor of up to six times. In all cases the increased radiation was accompanied by quite considerable cooling of the flames from approximately 1150°K to 900°K. The absolute temperatures as measured by thermocouple must be open to doubt but the trend will be correct, of course.

Ray and Long (61) determined the effects of various additives and changes in air flow on a commercial 'propane'-air diffusion flame. In fact the 'propane' contained 63% propylene but the significance of this has not been investigated. The soot from the flame was washed with pentane and the remainder referred to as carbonaceous residue. The polycyclic aromatic content of the soluble material was also measured. Increasing the air flow rate caused a drop in both soot and carbonaceous residue formation. Dichloro-methane increased the soot and carbonaceous residue levels in the flame but large amounts were required to do so, 60% w/w of the propane flow to double the soot concentration. The pentane-soluble material was not affected. Nitroethane, 1-nitropropane, ethyl nitrate and t-butylhydroperoxide all caused a reduction in the soot formation though the carbonaceous residue contents were unchanged. These results would be expected from the oxidising nature of the additives.

The effects of changing the fuel: oxidant ratio in premixed

methane-perchloric acid flames burning in an atmosphere of argon were examined by Pearson (101). Methane concentrations from 2 - 18 times stoichiometric were used. He found that in all cases only two molecules of methane per molecule of perchloric acid were reacted, the rest acting solely as a diluent. No carbon formation was observed in these flames, probably due to insufficiently high temperatures, details of which are not available. Oxygen was added to the flames and carbon formation was then detected. The oxygen may well have raised the temperature sufficiently to cause cracking of the methane.

Scully (102) injected an additive plus nitrogen axially into the products of combustion of a 10% rich town gas-air flame on a Meker burner. The soot produced was collected and measured.

33½% v/v benzene gave five times more soot than 3½% v/v acetylene butusing the same masses of benzene and acetylene, i.e. 100% v/v of the latter, resulted in equal quantities of soot formed, the aromatic ring apparently making no difference. Chlorobenzene however produced over twice as much soot as benzene, possibly due to the formation of hydrogen chloride and consequent polymerisation of phenyl radicals. Methyl groups attached to the benzene ring helped in a similar way but hydrogen gas inhibited carbon formation. The production of soot by the use of these aromatics depends on the additives themselves being converted to soot rather than causing part of the fuel supply to be degraded.

The luminescence of some common phospors and oxides of calcium in hydrogen and methane flames have been studied by

Mason (103). He observed that in a hydrogen flame luminescence occurred both when the flame impinged on a phospor spread on a support and also when the gases flowed through a porous body and burned on the coated surface. When methane was used luminescence was found only with the first arrangement. Phosphors were not added to the flame as part of the feed but it seems unlikely that the luminescence would occur over a wide enough band of the spectrum to usefully increase the radiation from the flame.

It is interesting to note the work of Blake et al (104) who measured carbon deposition from irradiated methane. The work was conducted at pressures 0.164 - 2.58 cm Hg, up to 10% of the methane being reduced to carbon depending on the dose of radiation. It was thought that the carbon was formed by way of ethane, propane, iso-butane and n-butane. Addition of carbon dioxide in large quantities considerably increased the deposition, up to 36%, there was no oxidation of the methane with production of carbon monoxide as had been expected. The addition of oxygen to the methane reduced the carbon formation at low dosages of radiation but had little effect at the higher rates. The pure methane and methane-carbon dioxide mixture experiments were repeated in the presence of graphite, higher yields were obtained in all cases. The carbon was preferentially deposited on the graphite rather than on the glass walls of the vessel being used. In the presence of graphite the levels of the saturated hydrocarbons were reduced and ethylene and acetylene were present in quantity. Similar results for atmospheric pressure would be useful.

During his work on the inhibition of methane and acetylene

diffusion flames, Miller (105) studied the effects of the addition of carbon tetrachloride, iron pentacarbonyl and chromous hypochlorite. Additive concentrations of less than 0.5% v/v were used, all three were observed to promote soot formation, carbon tetrachloride rather less so than the other two. Miller used additive concentrations much higher than those of Bonne et al (97) who did not find carbon formation when using iron pentacarbonyl. It is also significant that Miller used a diffusion flame whereas Bonne used a premixed flame.

Extensive work on the formation of soot and polycyclic aromatic hydrocarbons in ethane-air and ethylene-air diffusion flames has been carried out by Chakraborty and Long (62). In both cases the addition of oxygen to the fuel supply caused an initial increase in the amount of soot formed, a maximum was detected at the oxygen: ethane molar ratio of 0.62, corresponding to 18% of the stoichiometric oxygen. The soot formation at ratio 0.95 was almost the same as with no additional oxygen. The addition of hydrogen to ethylene flames caused a reduction in the soot formation but the effect on ethane flames was not measured. Adding methanol to the ethylene flame resulted in a steady decrease in the soot formed, a methanol:ethylene ratio of 1.0 reduced the soot by a third, the decrease being approximately linear with the methanol rate. Chakraborty and Long suggest that the methanol provided, through its own oxidation, species which either remove or prevent the formation of an intermediate in the production of soot.

Following the work of Homann et al (71, 73) Fenimore and Jones (106) compared premixed ethylene-oxygen and ethylene-nitrous oxide flames. They found that the latter produced eight times more soot than the former. Flame temperatures and concentrations of acetylene, diacetylene, triacetylene and hydrogen radical were similar in both types of flame. It was found that the concentration of oxygen and hydroxyl and oxygen radicals rapidly became the same although clearly the oxygen flame must be rich in oxygen radicals since it is known (107) that oxygen in hydrocarbon flames reacts according to

$$H + O_2 = OH + O$$

No reaction for the formation of oxygen atoms is known in nitrous oxide-hydrocarbon flames. It seems likely, therefore, that acetylene, polyacetylenes and their radicals which form early in the oxygen flame are being oxidised immediately by oxygen atoms. The soot formation is in fact promoted by the shortage of oxygen rather than the presence of nitrous oxide. Fenimore and Jones further found that adding hydrogen chloride to acetylene oxygen flames increased the green emission of C<sub>2</sub> and also the yellow luminosity. 60% w/w hydrogen chloride added to the acetylene trebled the yield of soot.

The effects of several inert additives on the smoke point of an ethylene-oxygen diffusion flame were measured by McLintock (108). All the smoke points were increased, i.e. the luminosities were decreased, and in order of effectiveness the additives were carbon dioxide, water vapour, nitrogen, helium and argon. It was found that the helium could be made to have a greater effect than the nitrogen by decreasing the size of the fuel nozzle relative

to the oxygen port. This phenomenon has not been explained but is probably due to the different gas flow rates and therefore gas velocities and the different rates of propagation of the flames producing dissimilar flame fronts and temperature profiles.

Dearden and Long (109) worked on flat diffusion flames of ethylene-air and propane-air. They found that raising the oxygen content of the air caused an increase in the sooting rate, ethylene producing more soot than propane. With ethylene a maximum sooting rate was not reached at an oxygen: ethylene molar ratio of 0.26 as Chakraborty and Long (62) had encountered. The discrepancy was probably due to the difference in burners, Chakraborty and Long used a cylindrical type. The addition of nitrogen to the fuel stream of the ethylene flame caused a decrease in soot formation, as did hydrogen in both flames studied. Acetylene caused a large increase in the soot formed in both. Oxygen in the fuel increased the sooting of the ethylene flame but slightly decreased that of the propane flame. Methane and acetylene concentrations were measured in both flames. In the ethylene flames with oxygen and acetylene as fuel additives it was shown that the sooting rate increased linearly with the quantity of acetylene present in the luminous zone. suggesting that acetylene is a stable intermediate during soot formation. The acetylene concentration fell sharply between the luminous region and the main reaction zone and so must be the main hydrocarbon oxidised in the latter. When oxygen was added to the propane the acetylene concentration was increased in the luminous region as it was in the ethylene flame. On this occasion, however,

the sooting rate fell contrary to the above theory.

Tischer and Scheller (110) carried out an investigation of the effect of adding boron trichloride to small cyanogen-oxygen diffusion flames. They found that the spectral intensity could be increased up to eight times in the range 0.44 - 0.67 and by consideration of their own results and those of Kaskan et al (111, 112, 113) concluded that the emission was due to boron dioxide. Since the increase in luminous intensity is restricted to such a narrow portion of the electromagnetic spectrum it is unlikely that the addition of boron trichloride to a methane flame would significantly increase its radiation.

The influence of a variety of organic solvents on the emission of acetylene-air flames seeded with sodium, potassium, calcium or magnesium salts was studied by Tesarik (114). The effect of the different solvents varied with the metallic radical present in the flame but in general the largest increase in emission was caused by n-propanol followed by acetone, iso-propanol, ether, methanol, ethanol, pyridene, water, ethylene glycol and tri-ethylamine. Only the emissions of the radicals, in narrow bands of the spectrum were measured and these cannot be related to the overall emissivities of the flames.

It is apparent that there has been a considerable amount of work done on additives in flames. Many types of flames have been studied and a wide variety of parameters have been measured but little has been published about the emissivity, radiance or even the temperature of flames with additives.

As part of his work in this field Comerford (125) used the method of Kurlbaum (126) to determine the spectral emissivity at 0.65 of propane diffusion flames. Propane and nitrogen were preheated in a furnace and partial pyrolysis took place before the fuel was burned in a stream of air. Comerford used the spectral emissivity as a measure of the soot concentration, however the theoretical treatment in Appendix 1 and considerable experimental evidence (21, 37) have shown that the emissivity of a cloud of particles is a function of the size of the particles, the path length and the particle density in addition to the mass concentration of the particles. No account of these factors was taken by Comerford, who found that the majority of the soot formation occurred in the flame rather than the pyrolysis tube. The length of the luminous zone was dependent on the air/fuel ratio rather than the velocity of the fuel. Since with high air/fuel ratios a definite point was reached along the flame where carbon combustion was completed and the soot zone had a sharp boundary, Comerford concluded that the burning of the soot was controlled by diffusion of air rather than the rate of reaction. With a limited air supply Comerford stated that the carbon concentration remained approximately constant after all the air was consumed despite having observed a gradual decline in the emissivity.

In 1961 Wurster et al (127) compared the luminosities of flames of several liquid and gaseous organic compounds using a 'luminometer'. The fuel was burned as a diffusion flame and the fuel rate adjusted so that the radiation was fixed at some arbitrary, constant level determined by a photocell. A highly luminous flame, therefore, burned at a slower rate than a less luminous one. A thermocouple was positioned above the flame, the

district and the

indicated temperature rising as the flame height increased.

The difference between this temperature and ambient was assumed to be in some way inversely proportional to the luminosity. Any differences between the flame speeds and adiabetic flame temperatures of the different field were not taken into account. The 'luminometer number' was defined

IN liquid = 
$$\frac{T_{\text{test fuel}}}{T_{\text{iso-octane}}}$$
 -  $\frac{T_{\text{tetralin}}}{T_{\text{tetralin}}}$  .100

IN gas =  $\frac{T_{\text{test fuel}}}{T_{\text{ethylene}}}$  -  $\frac{T_{\text{acetylene}}}{T_{\text{acetylene}}}$  .100

Since the characteristics of the photo-cell used are unknown and because of the simplifications made the 'luminometer number' is unlikely to be of significance.

The aim of this work is to provide information on the effects of a variety of additives on the radiating properties of flames. The information obtained is intended to be of use in predicting the performance of larger industrial-sized flames using similar additives. To this end it is intended to report on the effects of the additives on the important properties of the flames, rather than make a more fundamental investigation of the changes in reaction mechanism, within the flame, caused by the use of each additive.

# 5. Theoretical treatment of the proposed investigation

With the hope of finding an additive which would tend to increase the emissivity of a natural gas flame it was proposed to carry out an investigation into the influence of a range of additives on the radiation and associated properties of a premixed methane-air flame. Methane was chosen as the fuel since it comprises over 90% of the natural gas found in the North Sea and was preferred to the latter because it was available with constant composition.

Dependant variables to be measured were emissivity, flame temperature and radiance at different distances from the burner. Measurement of radiance alone was not considered to be of great value, the prediction of the radiance of similar flames of different size from this data being extremely difficult if not impossible. The radiance at any point in a flame may be considered as a function of the emissivity and temperature at that point. Considerable information is available about the scaling of emissivity of flames and temperatures can be predicted with reasonable accuracy by heat balances. It was felt, therefore, that measuring the emissivity and temperature would be of value.

In addition a measure was to be made of the heat transferred from the flame to its immediate surroundings, in this case a water-cooled calorimeter. It was hoped that the calorimeter would to some extent simulate the load in a furnace and some correlation between the temperatures and emissivities and the heat transferred might be obtained. The smoke points of the different flames, as

defined by Street and Thomas (85), were also to be determined to permit comparison with earlier work.

In order to reduce the effects of the experimental error involved in the tests and to improve the accuracy it was decided to carry out each run three times.

## 5.1 The measurement of emissivity, temperature and radiance

To reduce the amount of work it was decided to try to find one experimental method which would allow the determination of these three variables. The choice had to be made from the Schmidt method (115), the two path method and the two colour method, both described by Hottel and Broughton (29). The Schmidt method was rejected because of the difficulties outlined in section 3.2. Some of these problems were circumvented by Pengelly (116) who used a modified total radiation pyrometer. The method requires three values for radiation received by the pyrometer to be obtained. These are for the pyrometer sighted though the flame onto a cold, black backgroud, R<sub>F</sub>, through the flame onto a hot, black background, R<sub>F+B</sub>, and directly onto the hot, black background, R<sub>B</sub>.

Then, 
$$R_{F+B} = R_F + R_B \cdot \succeq_F$$

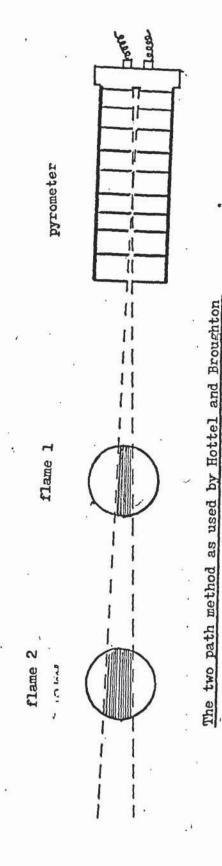
whence E provided that Kirchoff's Law may be assumed to hold, as discussed previously. Pengelly's instrument was fitted with two thermopiles which could simultaneously be sighted through the flame onto hot and cold, black backgrounds.

The two colour method as conceived by Hottel and Broughton

required the use of two pyrometers operating in different, extremely narrow bands of the spectrum. The pyrometers were calibrated against a black body and the readings obtained when sighting them at flames were referred to as 'colour temperatures'. From the two temperatures at the same position in the flame Hottel and Broughton were able to evaluate the true flame temperature and the emissivity. To perform this calculation, however, it was necessary to assume a value for the absorption coefficient of the soot. An average value was used but the evidence against the validity of the assumption, on the grounds that the coefficient is a function of the particle size, is considerable (117). Variations of the method have included the use of an optical pyrometer for measuring the red brightness temperature together with a total radiation type (118, 119) and the three colour method (120). In both cases it is still necessary that the absorption coefficient is known. The only alternative to assuming a value is to obtain the particle size and size distribution for the soot and evaluate it.

The two path method could be considered as a modification of the Schmidt method. The main disadvantage of the latter, that to avoid assuming the flame is a non-grey body then the hot background must be at the flame temperature, is circumvented by using an identical flame as the hot background. The background is then automatically at the correct temperature.

The two path method was recommended by Dyne and Penner (121) for use in situations in which spectral emissivity data are not available. It has to be accepted, however, that the emissivity



pyrometer flame plane mirror

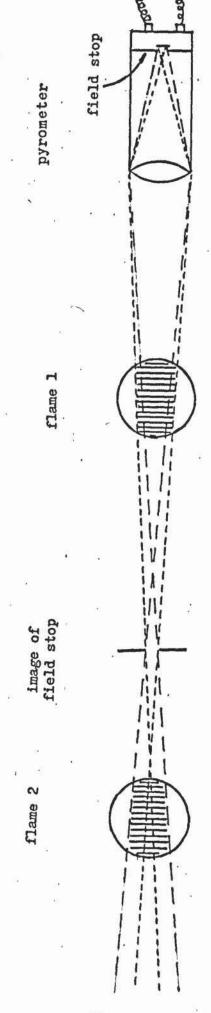
Figure 10

The two path method using a mirror

17

11

11 11



The two path method using a lens-type pyrometer

(Not to scale)

over the range of wavelengths to be covered will be represented by a single average value. This restriction does not prove a disadvantage in an engineering investigation such as this, when it is wished to compare the emissivities of flames averaged over all wavelengths. It should be noted, however, that the measuring of average emissivities for the flames does not imply that they behave as grey bodies.

Hottel and Broughton used a diaphragm-type total radiation pyrometer viewing first one flame and then the same flame with an identical one beyond it, as shown in fig. 9. More recent applications (122, 123, 124) have used a mirror of known reflectance in place of the second flame as shown in fig. 10.

It is clear that a much larger area of the 'second flame' than of the first flame falls within the field of view of the pyrometer. If the flame is relatively large and areas within the field of view can be considered isothermal then errors resulting from viewing different areas of the two flames will be small. This is undoubtedly the case when the technique has been used for measuring temperatures in pulse jets and rocket chambers (122, 123). For use with small flames the errors may be much more serious and the values for emissivity and temperature would become meaningless.

To overcome this problem it was decided to utilise a property of the field of view of the lens-type total radiation pyrometer which was available, fig. 11. It can be seen that the field of view converges up to the image of the field stop and beyond this point it diverges.

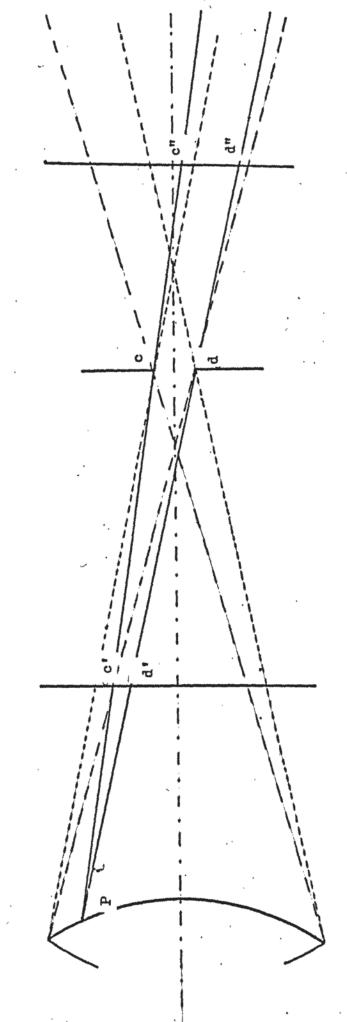
By positioning one flame in each part of the field of view almost exactly similar portions of the two flames were detected and errors due to assumptions of isothermal regions were minimised.

#### 5.2 The Pyrometer

## 5.2.1 The field of view of the pyrometer

The total radiation pyrometer used, which is described in more detail in section 7.3, had a lens of diameter 3.5 cm, focal length 5.5 cm and the field stop was of 0.4 cm diameter, positioned 8 cm from the lens. The lens was of the convexconvex type and was 4.67 mm thick at the axis so the position and size of the image of the field stop could be found by use of the thin lens formulae. The image was 17.606 cm from the lens and was of 0.88 cm diameter.

The field of view of the pyrometer was drawn out as shown in fig. 11. It should be noted that due to the finite size of the field stop the two sections of the field of view converge and diverge at slightly different angles. This was assumed to be insignificant when the positions of the two flames were selected, equal diameters in the field of view at the centre line of each flame being the criterion used. The differences in volume and shape of the portions of flame viewed were very small and considered to be of little consequence; but rather than calculate the resultant error, which would have been extremely difficult, a preliminary experiment was carried out to obtain suitable correction factors. This is described in sections 5.2.4 and 8.1.



To show that theoretically the radiation received by the pyrometer is the same from both flames

Pigure 12

flame 2

image of field stop

flame 1

pyrometer lens

-63-

It was decided to position the flames at 14.1 cm and 19.6 cm from the lens of the pyrometer. The field of view at both these points was of 1.4 cm diameter.

## 5.2.2 The radiation received by the pyrometer

Considering the situation shown schematically in fig. 12 with a pyrometer viewing a large flame, the radiation reaching the thermopile must pass through the lens aperture, and depending on the relative positions of pyrometer and flame, must either pass through or appear to pass through the image of the field stop that would be formed by the lens. The only part of the flame radiating to the thermopile via point P on the surface of the lens is that represented in flame 1 by c'd' or in flame 2 by c"d". If A is the area of the image of the field stop and A' and A" the areas of c'd' and c"d" respectively then by similar triangles

$$\frac{A}{(Pc)^2} = \frac{A!}{(Pc!)^2} = \frac{A"}{(Pc")^2}$$

by the Inverse Square Law

irradiance of P by flame 1 
$$\frac{A^{1}}{(P^{c})^{2}}$$

and irradiance of P by flame 2 
$$\propto \frac{A''}{(P''c'')^2}$$

If flames 1 and 2 are similar, therefore, then the irradiance of P, by radiation which will pass to the thermopile,

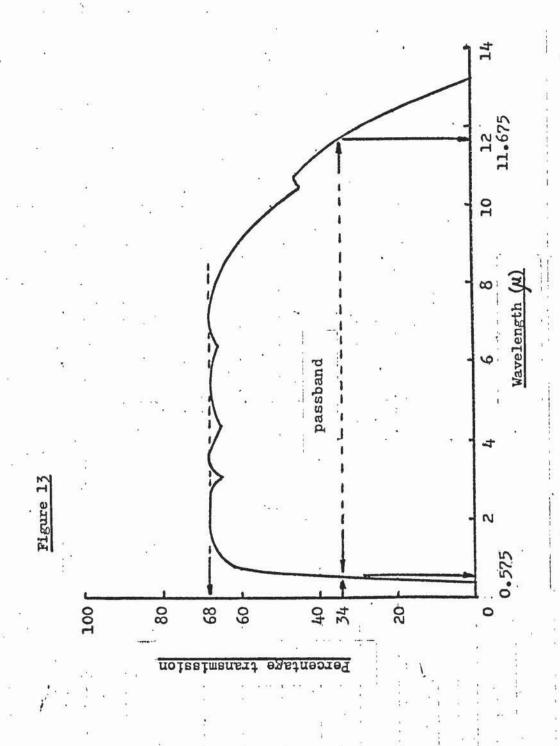
will be the same in both cases and, indeed, for similar flames at any distance provided they fill the field of view. This argument may be extended to all points on the surface of the lens and so in theory the radiation reaching the thermopile is independent of the distance of the pyrometer from the flame.

From this it may be concluded that identical readings should be obtained for the two identical flames located at 14.1 cm and 19.6 cm from the pyrometer lens. In practice atmospheric absorption of radiation by water vapour might occur though the loss would be relatively small. It is possible to calculate corrections for this phenomenon from a knowledge of the humidity and temperature (128). For the purpose of this work it was considered satisfactory to use a constant correction factor, determined experimentally and incorporated into the correction for field of view mentioned previously and detailed in Section 5.2.4.

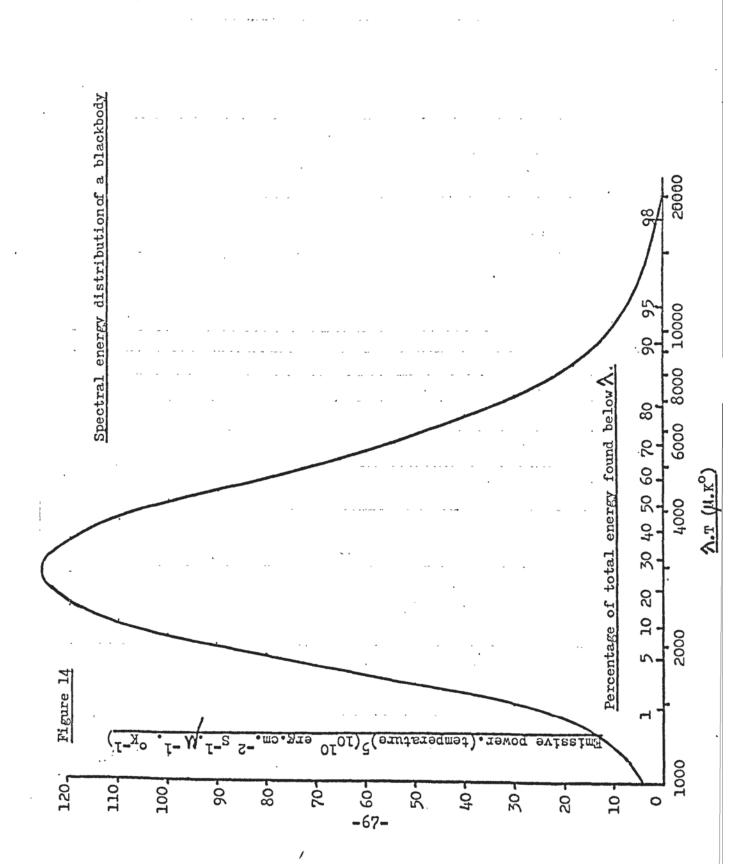
#### 5.2.3. The passband of the pyrometer lens

The main disadvantage of lens-type radiation pyrometers, such as that used for the present work, is that radiation of all wavelengths is not transmitted equally through the lens material. It is not strictly accurate, therefore, to state that such an instrument measures total radiation, some absorption and reflection of radiation is unavoidable to an extent which varies with wavelength.

Daws and Thring (33) pointed out that the radiation of significance in most heat treatment furnaces had wavelengths within the limits 1 - 7 to fulfil the aims of the present work



The passband of arsenic trisulphide 3mm thick



and provide meaningful values for the emissivities of the various flames examined it was, therefore, necessary to detect the radiation from the flames within those limits of wavelength. Several different lens materials have been used for investigations in this region of the spectrum, among them are fluroite and arsenic trisulphide. The latter was considered to be most suitable since calcium fluoride lenses are subject to extremely bad chromatic aberration leading to difficulty in defining the field of view. A plot of percentage transmission against wavelength for arsenic trisulphide is shown in fig. 13. The refractive index in the range shown is approximately 2.4, giving a reflection loss for two surfaces of about 30%. The reflection loss could have been reduced to about 5% by the use of anti-reflection coatings; this would have increased the output voltage of the pyrometer at any particular target temperature but would not have significantly extended the range of radiation detected. As the output of the pyrometer was considered sufficient the anti-reflection coatings were not used.

The limits of the pass band of a lens are defined as those wavelengths at which the transmission is 50% of the maximum. From fig. 13 it can be seen that the passband of an arsenic trisulphide lens is 0.575 - 11.675 \mu, which more than fulfils the requirements of Daws and Thring. The distribution of energy in the spectrum of a black body is shown in fig. 14 (129), only 3% of the total energy radiated is of wavelengths outside the passband of the arsenic trisulphide lens. The soot particles in the flames may be considered approximately as black bodies, see Appendix 1, and

so the distribution of spectral energy will be similar to that in fig. 14. While the approximately 3% unmeasured energy has to be taken into account, any inaccuracy due to measuring radiation in a restricted portion of the spectrum will be very slight.

#### 5.2.4. The calibration of the pyrometer

The basic equation for the output of the pyrometer may be written as

output < (radiance of the source).(transmission factor)

The radiance of the source is given by the Stefan-Boltzmann law

$$W = \epsilon . \sigma. T^4$$

The transmission factor is made up of two parts, one being a function of the pass band of the lens and compensating for the radiation not detected. The other part is a function of the temperature of the source and accounts for the variation of the spectral energy distribution with the temperature of the source. The monochromatic radiance can be seen to be a function of temperature by reference to Planck's law for a black body,

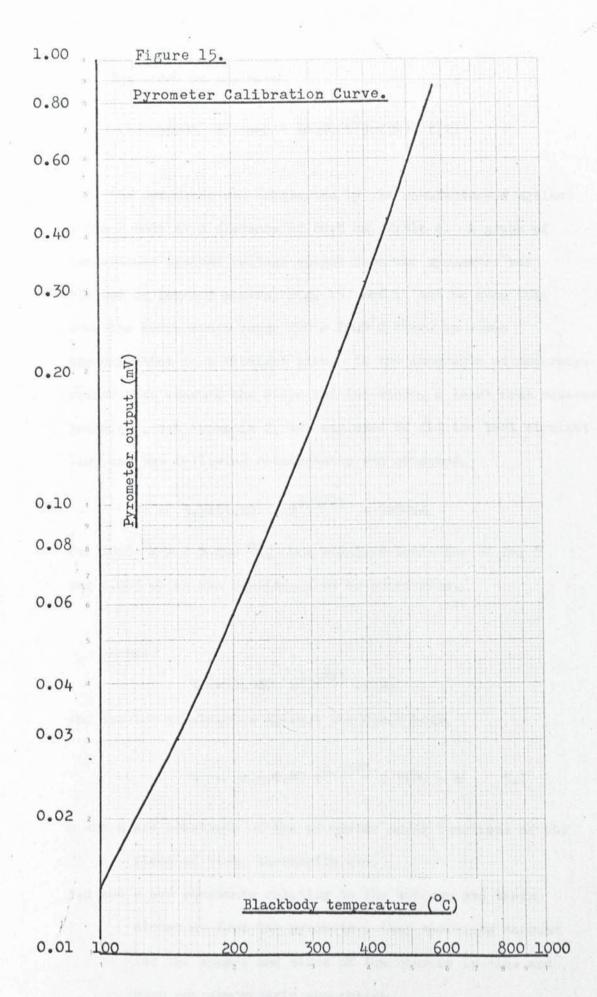
$$W_{B} = \frac{c_{1} \cdot \lambda^{-5}}{\exp(\frac{c_{2}}{\lambda T_{B}})^{-1}}$$

This correction is, again, necessary because the pyrometer only detects radiation in a limited portion of the spectrum.

Table 4

The Pyrometer Calibration data

Temp.(°C)	Emf.(mV)	Temp.(°C)	Emf.(mV)	Temp.(°C)	Emf.(mV)
100	0.014	400	0.315	700	1.615
110	0.017	. 410	0.337	710	1.688
120	0.020	420	0.361	720	1.761
130	0.023	430	0.385	730	1.837
140	0.026	440	0.410	740	1.917
150	0.030	450	0.438	750	1.999
160	0.035	460	0.466	760	2.083
170	0.040	470	0.494	770	2.169
180	0.045	480	0.525	780	2.260
190	0.050	490	0.557	790 .	2.353
200	0.057	500	0.591	800 ,	2.447
210	0.064	510	0.626	810	2.542
220	0.071	520	0.662	820 .	2.640
230	0.079	530	0.700	830	2.742
240	0.087	540	0.740	840	2.852
250 .	0.096	550	0.780	850	2.964
260	0.105	560	0.823	860	3.078
270	0.115	570	0.867	870	3.195
280	0.126	580	0.913	880	3.315
290	0.137	590	0.961	890	3-438
300	0.149	600	1.011	900	3.563
310	0.161	610	1.063	910	3.691
320	0.174	620	1.115	920	3.823
330	0.189	630	1.170	930	3.958
340	0.204	640	1.227	940	4.095
350	0.220	650	1.287	950	4.237
360	0.237	660	1.349	960	4.383
370	0.255	670	1.412	970	4.533
380	0.274	680	1.475	980	4.685
390	0.294	690	1.543	990	4.841
400	0.315	700	1.615	1000	5.000



Therefore we now have

output 
$$V \ll (\epsilon.\sigma. T^4).f(A).f(T)$$

The pyrometer was calibrated by the manufacturers against a black body at a distance of 20.3 cm, Table 4. A graph of temperature against voltage output from the pyrometer was plotted on log-log scales, fig. 15, and it can be seen that over the temperature range 200 - 1000°C there is close approximation to a straight line. In the interests of accuracy, rather than measure the slope and intercept, a least mean squares technique, see Appendix 2, was employed to fit the best straight line and the following relationship was obtained,

$$V = 3.5481.10^{-4}. T^{4.5773} + 0.0024$$

for 1000  ${}^{\circ}$ K > T > 200  ${}^{\circ}$ K. The standard deviation of log T was 0.003 which was considered to be acceptable.

Hence

and for the calibration against the black body,

$$V = m.p. \epsilon. \sigma. T^{4.5773} . f(\lambda) + n$$
 5.1

m and n are constants of the pyrometer being functions of the field of view, thermopile etc.

p,q and r are constants relating to the sources and their distances from the pyrometer, they therefore account for the shapes and sizes of the sources in view and also any atmospheric absorption.

p is for a source at the calibration distance of 20.3 cm.

q and r are for sources at 14.1 cm and 19.6 cm respectively.

It is important to realise that for a non-black source the temperature indicated by the pyrometer will not be its true temperature, but the temperature of a black body which would cause the pyrometer to give the same output, this is referred to as the brightness temperature of the source, T<sub>b</sub>. Therefore in addition to equation 5.1

$$V = m \cdot p \cdot \sigma \cdot T_b^{4.5773} \cdot f(\lambda) + n$$
 5.2

For a flame of true temperature T which has a brightness temperature  $T_{\rm p}$ , measured at the calibration distance,

$$V_p + m.p.e.s. T^{4.5773}.f(\lambda) + n = m.p.s.T_p^{4.5773}.f(\lambda) + n$$

whence

$$\epsilon \cdot \mathbf{T}^{4.5773} = \mathbf{T}_{\mathbf{D}}^{4.5773}$$
 5.3

Theoretically the brightness temperature should be independent of the distance from the pyrometer provided that the field of view is filled. However, because of the reasons detailed in sections 5.2.1 and 5.2.2 this may not be the case.

For the same source having a brightness temperature  $T_Q$  measured at 14.1 cm,

$$V_Q = m.q.e.at^{4.5773}.f(\lambda) + n = m.p.s. T_Q^{4.5773}.f(\lambda) + n$$

Whence

$$q.\epsilon. T^{4.5773} = p.T_Q^{4.5773}$$
 5.4

For the same source having a brightness temperature  $T_{\rm p}$  measured at 19.6 cm,

$$V_{R} = m.r.e.s.T^{4.5773}.f(\lambda) + n = m.p.s.T_{R}^{4.5773}.f(\lambda) + n$$

whence

$$r. \in T^{4.5773} = p.T_R^{4.5773}$$
 5.5

From equations 5.3 and 5.4

$$\frac{p}{q} = \left(\frac{T_p}{T_Q}\right)^{4.5773}$$

From equations 5.4 and 5.5

$$\frac{q}{r} = \left(\frac{T_Q}{T_P}\right)^{4.5773}$$

Since the pyrometer constants, m and n, could be eliminated in every case as above their values were not required. As will be shown in the next section the absolute values of p, q and r were also not necessary. The two ratios p/q and q/r were calculated from equations 5.6 and 5.7 after a simple preliminary experiment in which the brightness temperatures of identical flames were measured at the three distances, see section 8.1.

# 5.2.5 The calculation of emissivity, temperature and radiance

For a flame, true temperature T, emissivity  $\epsilon$  and brightness temperature  $T_1$  measured at 14.1 cm,

$$V_1 = m.q. \epsilon. \sigma. T^{4.5773}.f(\lambda) + n = m.p. \sigma. T_1^{4.5773}.f(\lambda) + n$$

Whence

$$\in \cdot q \cdot T^{4 \cdot 5773} = p \cdot T_1^{4 \cdot 5773}$$
 5.8

For two identical flames, true temperature T, emissivity e, combined brightness temperature T<sub>2</sub> measured with the pyrometer viewing one flame at 19.6 cm through the other at 14.1 cm,

$$V_2 = m \cdot f(\lambda) \cdot T^{0.5773} \cdot (q.\epsilon.\sigma. T^4 + c.r.\epsilon.\sigma. T^4) + n$$
  
=  $m \cdot p \cdot f(\lambda) \cdot \sigma. T_2^{4.5773} + n$ 

Whence

$$\epsilon \cdot q \cdot T^{4 \cdot 5773} + \epsilon \cdot r \cdot T^{4 \cdot 5773} = p \cdot T_2^{4 \cdot 5773}$$
 5.9

From equations 5.8 and 5.9

$$1 + \varepsilon \cdot \frac{r}{q} = (\frac{T_2}{T_1})^{4.5773}$$
 5.10

By Kirchoff's law when the source of the transmitted radiation is at the same temperature as the transmitting body, as in this case

From equations 5.10 and 5.11

$$\epsilon = 1 + \frac{q}{r} - \frac{q}{r} \left( \frac{T_2}{T_1} \right)^{4.5773}$$
 5.12

By rearranging equation 5.8

$$T = \left(\frac{p}{\epsilon \cdot q}\right)^{1/4 \cdot 5773} \cdot T_1 \qquad 5.13$$

By the Stefan-Boltzmann law

$$W = \frac{\epsilon.\sigma. T^4}{\pi}$$
 5.14

## 5.3 The smoke point

The definition of smoke point to be used was similar to that of Street and Thomas (85) being the critical quantity of air required to suppress carbon formation. The critical air flow was a mean of the air flows when yellow luminosity could just be observed all the time and when the yellow luminosity had just disappeared. The flow was to be expressed as a fraction of the stoichiometric air for the methane, no allowance being made for the additive.

## 6. Plan of the experimental work

#### 6.1 The methane flow

As previously stated methane was selected as the fuel to be used since it was available with constant composition. Natural gas was not available from the local gasboard and so would have had to have been obtained in cylinders, as did the methane. The natural gas would not have had a fixed composition which was considered a serious disadvantage when comparing the effects of additives in small quantities.

The methane used had composition

CH<sub>4</sub> 99.9% +

0<sub>2</sub> less than 10 v.p.m.

CO, less than 5 v.p.m.

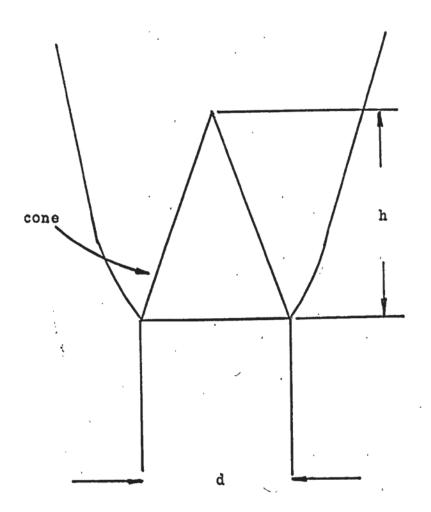
N<sub>2</sub> balance

The methane was supplied in cylinders of 623001 measured at S.T.P., capacity at a pressure of about 102 atm. For reasons of cost and also to reduce delays caused by changing cylinders it was necessary to use a small bench-scale flame rather than an industrial-sized flame. It was anticipated that a complete investigation of a single additive would take up to 5 hours. A methane flow of about 0.0264 1.s<sup>-1</sup> per burner was, therefore, equivalent to approximately 730 1 per additive.

## 6.2 Burners

Preliminary calculations of theoretical flame dimensions from burner size and air and methane flow rates were performed using the following equation:

Figure 16.



Idealised flame used to calculate approximate
dimensions at various flow rates

$$Q_f = 0.25. V_f \cdot \pi. d^2. (1 + (\frac{2h}{d})^2)^{0.5}$$

which is based on the idealised flame shown in fig. 16.

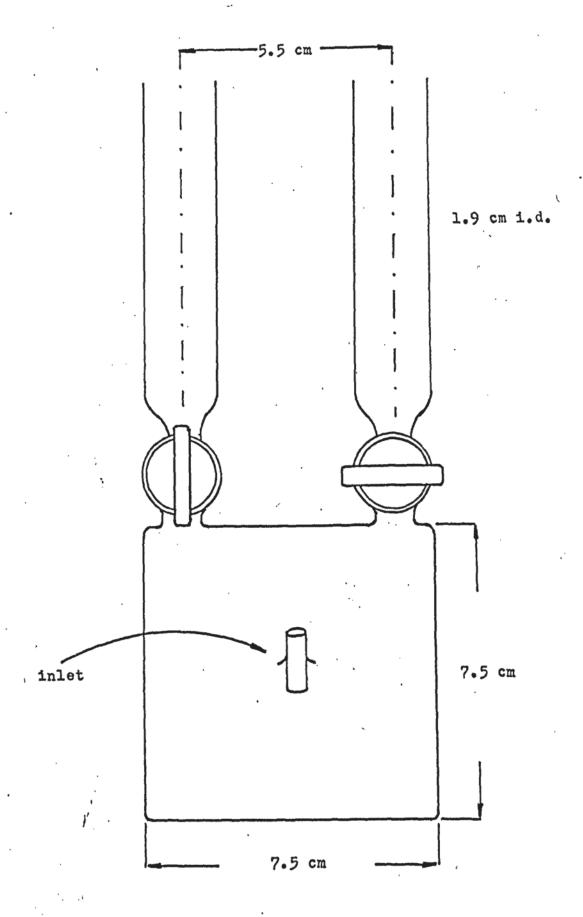
The results of these calculations, the requirement that the flames should fill the field of view of the pyrometer; and factors concerning the air supply to the flame which will be discussed in section 7.2, were all considered before arriving at the final choice of burner size and methane and air rates. Burners of 1.9 cm diameter were used, positioned with their vertical axes 14.1 and 19.6 cm from the lens of the pyrometer. The methane flow rate per burner was fixed at 0.0264 1.s<sup>-1</sup>, premixed with 50% stoichiometric air. This methane rate corresponded to eight additives per cylinder but in practice this was bettered as experimental technique improved.

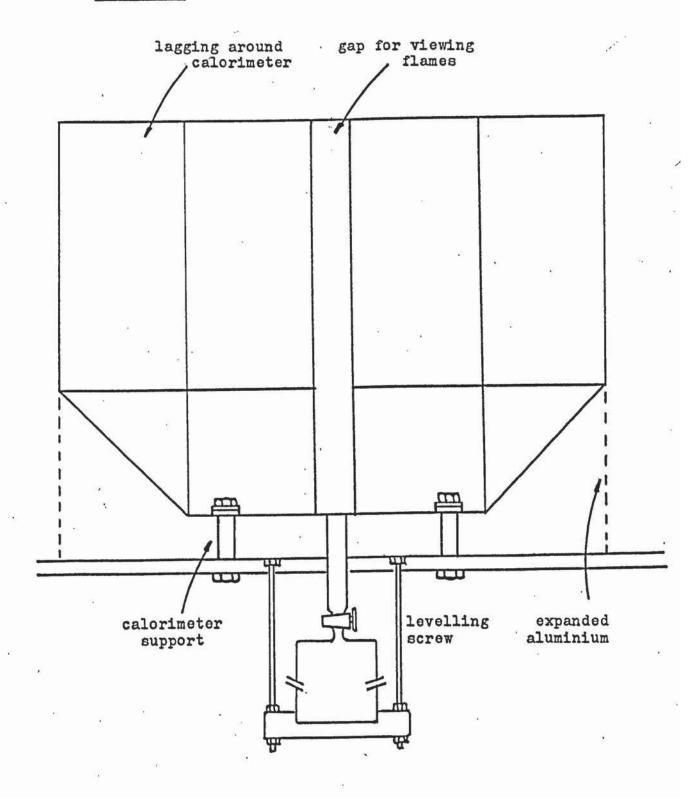
A preliminary test showed that the burner chosen gave
a flame of shape similar to that suggested by the calculations
and that all the conditions were fulfilled. The flame had a
visible height of about 10-12 cm and a cone height of about 6 cm.

#### 6.3 Additive flows

The relatively low methane flow caused difficulty because of the very low additive flows necessary. Although it would have been preferable to have used another lower additive flowrate, around 1% w/w of the methane rate, it was felt that the additive rates 2.5, 5, 10, 15 and 20% w/w of the methane rate adequately

covered the range of interest without creating too many problems.





Arrangement of the burners and calorimeter

### 7. The apparatus and equipment

### 7.1 The burners

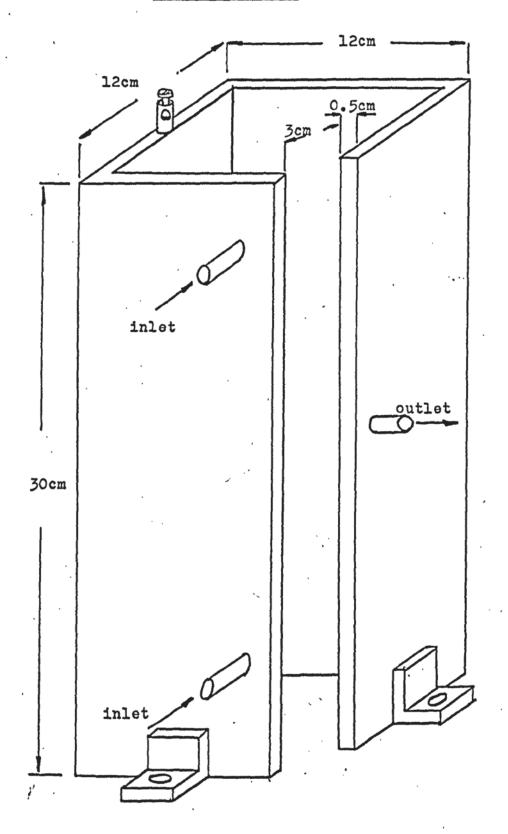
The burners, shown in fig. 17, were glass tubes 8 cm long and 1.9 cm internal diameter and were mounted on a glass mixing chamber. The methane was admitted via a tube in one side of the mixing chamber and the air plus additive, if any, entered through a similar tube opposite. The mixture was passed to one or both of the burners by way of stop-cocksat the bases of the burner tubes. It was expected that some positive means of mixing the methane with the air and additive, such as baffles, would have to be provided. Tests conducted on the burners showed, in fact, that they were not necessary and they were not used in the investigation of the additives, see section 8.4. A mesh of stainless steel wire was positioned just inside the top of each burner to act as a flame trap and prevent lighting back into the mixing chamber. The mesh was supported on a length of stiff wire standing on the stop-cock.

The mixing chamber was carried in a wooden frame hanging by three brass screws from the calorimeter base-board, fig. 18.

By adjusting the screws it was possible to alter the height of the mixing chamber and set the burners vertically, the latter projecting through the calorimeter base-board.

The mixing chamber and the lower parts of the burner tubes were wrapped with heating tape which was used to prevent condensation of the higher boiling point additives. Preliminary tests showed that the small amount of heating, 0.24 Kw, did not

The calorimeter.



affect the emissivity or temperature of the flames significantly see section 8.3.

#### 7.2 The calorimeter

The calorimeter was constructed from 16 gauge copper sheet, fig. 19, all joints being soldered. The internal dimensions of the calorimeter were dictated by the size and position of the flames. To reduce the size as much as possible the flames were positioned on a diagonal. It was intended that the flames would not impinge on the walls of the calorimeter. A height of 30 cm was chosen to accommodate the longest possible flame which might have been obtained by the use of additives. The gap of 3 cm in one corner through which the pyrometer viewed the flame was selected so that none of the radiation from the flames was intercepted A small vent was fitted to enable all the air to be bled from the calorimeter.

So that the warm up time of the calorimeter was not too long and to achieve a fairly quick response in outlet temperature to any change in the flame conditions it was necessary to keep the thermal capacity of the calorimeter low. This was achieved by positioning the inner and outer surfaces relatively close together, at 0.5 cm, and having a small volume of water in the calorimeter at any time.

The inner surface of the calorimeter was painted with a mixture of lampblack and waterglass (sodium silicate solution) which has an emissivity of 0.95 - 0.96 (130, 131). The effective

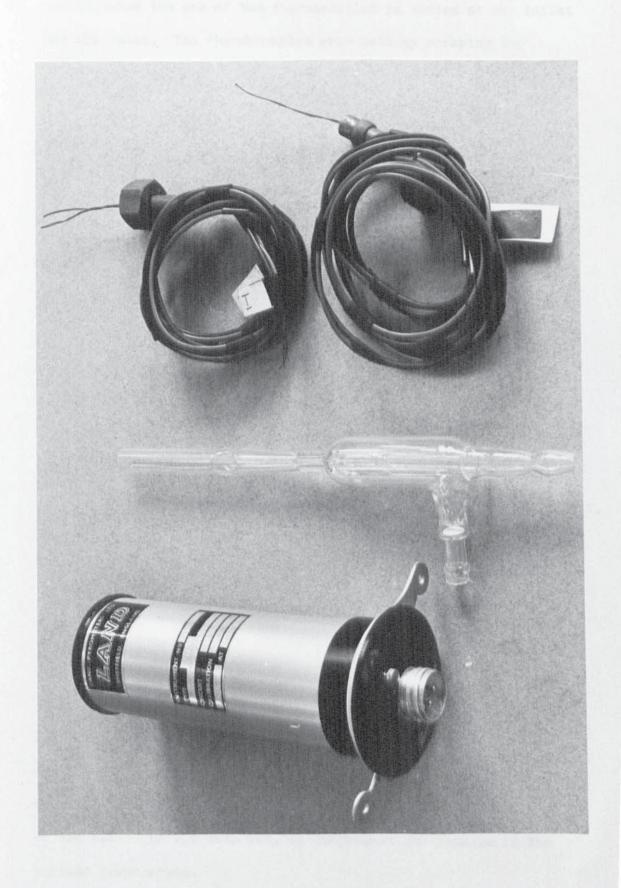
emissivity of the walls to the pyrometer was also brought closer to the unity because they were not normal to the axis of the field of view but at 45°. Approximately 1 g of lampblack was used per 4 ml water glass. It was necessary to apply the solution in a very thin coat, otherwise the glass would flow resulting in a shiny surface.

The outside walls of the calorimeter were lagged with a 15 cm thick layer of expanded polystyrene. Assuming a mean outside wall temperature of 45°C the rate of heat loss through the lagging was calculated to be approximately 0.2 cal.s<sup>-1</sup>. The heat supplied by the methane was approximately 236 cal.s<sup>-1</sup>, although only a fraction of this heat was transmitted to the calorimeter, the wall losses could be considered negligible. Losses due to radiation through the gap in the calorimeter could not be calculated without prior knowledge of the flame temperature and emissivity.

In order to promote good circulation of the water in the calorimeter and to avoid local hot spots, two water inlets were provided as shown in fig. 19. It was not possible to use two outlets also since this would have caused difficulty in obtaining the mean outlet temperature. A Fischer-Porter flowrator, FP - 1/8 - 20 - G, with a stainless steel float was used to measure the water flow which was maintained constant at 115 ml.m<sup>-1</sup>. The flowrator calibration was checked before and after the additive investigations, Appendix 3.

The temperature of the water was measured by chromel-alumel

Figure 20 Pyrometer, vacuum pump and thermocouples.



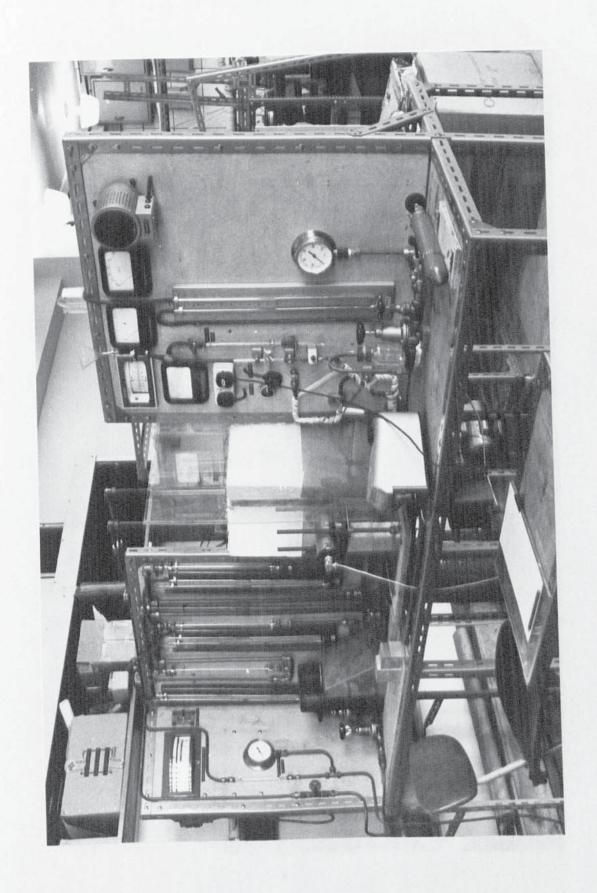
thermocouples. The relatively low temperatures to be measured necessitated the use of two thermocouples in series at the outlet and the inlet. The thermocouples were made by scraping the insulating lacquer from the wires and welding the ends together. They were calibrated against an accurate mercury-inglass thermometer at approximately 0°C and 100°C, their outputs being measured with a potentiometer, due accountwas made of the cold junction temperature. Those couples which did not satisfy the requirements of B.S.1827 were rejected and replacements made up. The thermocouples, singly or in pairs as required, were fixed into short lengths of 3/8 in. outside diameter copper tube using Araldite. The couples were so positioned that when the tubes were connected to the brass compression fittings at the inlets and outlet of the calorimeter their junctions were just inside the outer wall. Thermocouples in their tubes are shown in fig. 20.

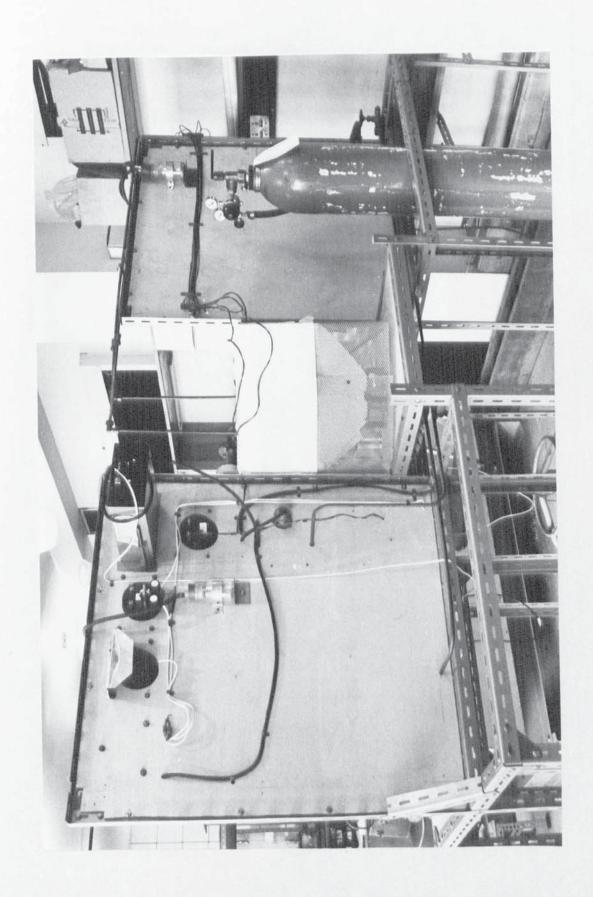
Temperature teadings were taken from a moving coil galvanometer, Electroflo Meters Co. Ltd. Model 1511, which had been specially calibrated for two chromel-alumel thermocouples in series and had a full scale deflection of approximately 8 mV. Changing from inlet to outlet temperature reading was by a Model 151/23 two-way switch with platinum contacts. All the wiring from the couples to the galvanometer was chromel or alumel, compensating leads were not used. The recalibration of the meter for use with two thermocouples meant that the automatic cold junction compensation would not be employed. It was necessary, therefore, to adjust the zero of the meter, before each run, to compensate for changes in the ambient temperature.

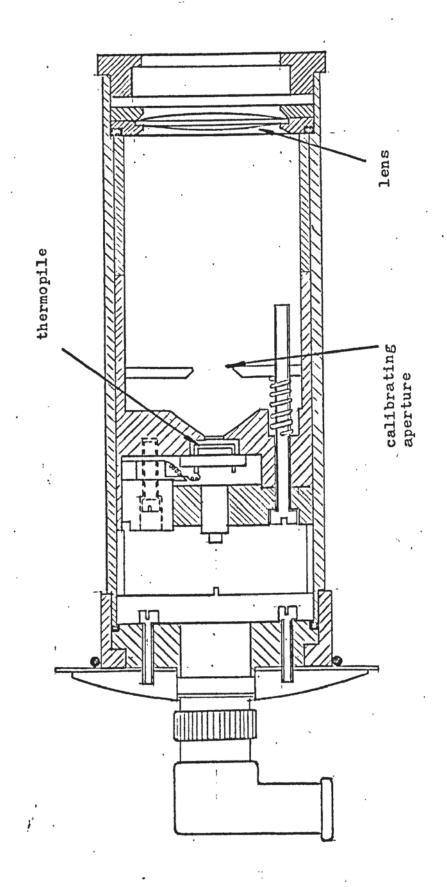
The gap in the walls of the calorimeter caused considerable difficulty during early tests, mainly because secondary air could not be excluded. Initially the calorimeter was mounted directly onto the base board so that air could not enter under the walls. In this position it was found that however much primary air was premixed with the methane, up to the onset of lift off, some air was being drawn in through the gap. Attempts to fill the gap above and below the pyrometer were not entirely successful and the nearer flame was distorted by the incoming air. A satisfactory solution was found by raising the calorimeter 4 cm from the base board and fixing the air flows at 50% stoichiometric. As a further safeguard a perspex shield was fitted beneath the pyrometer mounting jacket and this was sealed against the calorimeter lagging by strips of polyurethane foam.

In themselves these measures, allowing air under the calorimeter walls and reducing the suction through the gap, were not quite sufficient but due to draughts in the laboratory affecting the flames it was considered undesirable to raise the calorimeter further. In order to enable more air to pass under the walls the expanded polystyrene lagging was cut away from the lower edge of the calorimeter at 45°. The tops of the burners were level with the lower edge of the calorimeter. Viewed from above the flames showed a satisfactory circular section and there was no consistent tendency towards distortion.

The above modifications made the flames susceptible to draughts causing intermittant distortion. The gap between the outer surfaces







of the lagging and the base board were, therefore, filled using expanded aluminium mesh, as can be seen in figs. 21 and 22. This material has a large portion of its surface area available for air to pass through but, because of the twist imparted to the mesh as it was expanded most of the air was diverted from its original path and did not flow directly to the flame. The effects of draughts were thereby much reduced but as a further precaution one side of the rig was screened from the laboratory by hardboard panels.

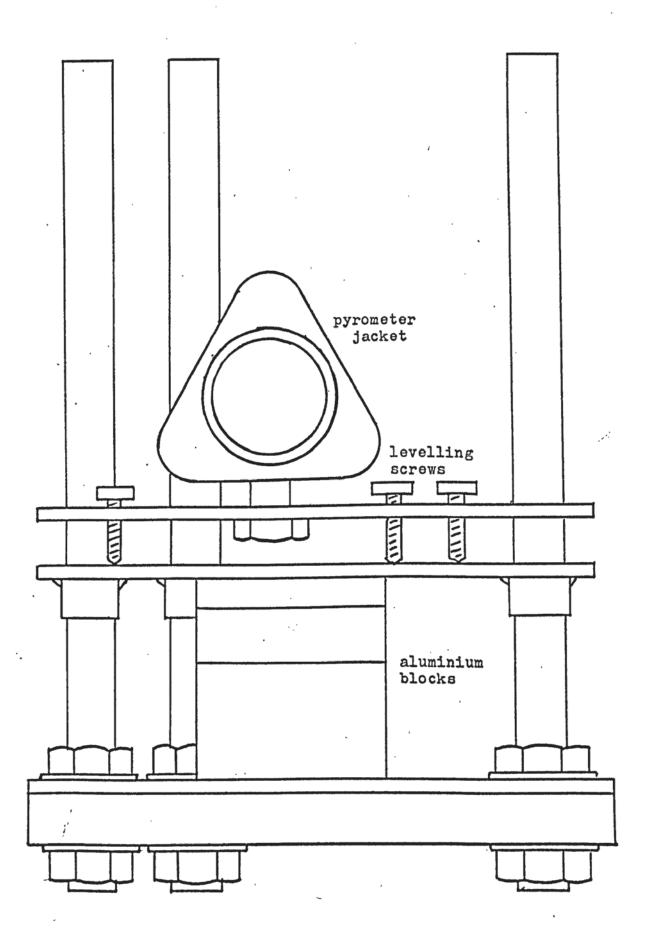
#### 7.3 The Pyrometer

The pyrometer used was supplied by Land Pyrometers Ltd.

Type ORF 35/20/6, see figs. 20 and 23. The lens was of arsenic trisulphide with an aperture of 35 mm, no protective window was used and the angular field of view was 0.05 radian.

The response time was approximately 2 s to 98% of the final output. The pyrometer was compensated so that variations of its temperature in the range 0°C to 100°C had a negligible effect on the output. This was achieved by shunting the thermopile with a temperature sensitive resistor having a value of 60 ohm at 20°C. The resistance of the thermopile itself was approximately 30 ohm whence the effective internal resistance of the pyrometer was given by

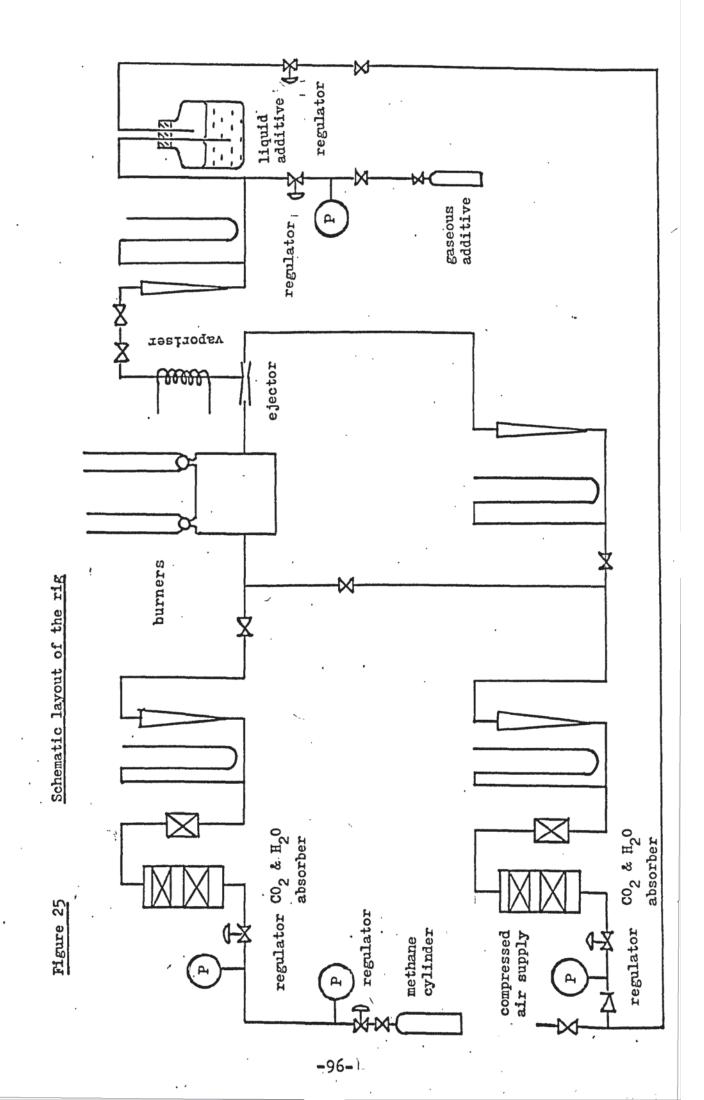
$$\frac{1}{R_{\text{pyro}}} = \frac{1}{30} + \frac{1}{60}$$



For the temperature compensation to be effective it was necessary to use a measuring instrument of resistance greater than 200 ohm. Otherwise the compensation would have been shunted and become less effective.

An Ether Type 1250 B potentiometric temperature indicator specifically calibrated for the pyrometer was available. Neigher of the two ranges provided, 0°C to 1000°C and 0°C to 1800°C, was sufficiently accurate for the comparatively low brightness temperatures encountered, however. Instead a Pye Scalamp moving coil galvanometer was used. The meter was supplied with an arbitrary scale and was calibrated using a Cambridge potentiometer, see Appendix 3. The pyrometer was connected to the galvanometer by copper leads of resistance approximately 1 ohm.

The pyrometer was housed in a cast aluminium jacket which acted as a heat sink. The jacket was fixed by a single bolt to a horizontal plate supported on an identical plate by three setscrews, fig. 24. The pyrometer could be levelled by adjusting the screws. The two plates were free to slide up and down three steel columns firmly mounted on a wooden base board faced with a sheet of aluminium. The plates and pyrometer were supported at various heights by blocks of aluminium stacked on top of one another. The blocks were machined with parallel surfaces at thicknesses of 1, 2, 4, 6 and 8 cm to an accuracy of 1/1000 in. Suitable combinations of blocks enabled any desired height of the pyrometer above the burners to be obtained quickly and accurately. The height adjustment was zeroed, by altering the position of the



calorimeter baseboard, such that using the 6 cm block alone placed the pyrometer axis level with the top of the burners. The baseboard was mounted so that it could be moved in both horizontal planes, towards and away from the burners and also laterally.

# 7.4 The methene supply

The layout of the various supplies is shown schematically in fig. 25. The methane cylinders were fitted with a Matheson No. 36 high purity gas regulator, the pressure of methane into the rig being maintained at 30 lbf. in 2g. The pressure was further reduced by a Norgren E2A regulator before the methane passed into a combined moisture and carbon dioxide absorber. This was probably unnecessary for the grade of methane used but consisted of a 30 cm long, 7.5 cm diameter mild steel vessel containing two packed beds, one being 200 g of silica gel and the other being 200 g of 'Carbosorb', a mixture of soda and asbestos containing 63% w/w Na20. The beds were supported on copper mesh fixed to the walls of the vessel.

After the absorber the methane flowed through a fritted glass filter to stop any particles of Carbosorb being carried through the pipework which was a combination of 3/8 in. and  $\frac{1}{2}$  in. o.d. copper tubing.

Metering of the methane flow was by a Metric 7X Rotameter having a duralumin float. The pressure in the rotameter was measured by a mercury manometer and was maintained at 10 cm Hg g. by the regulator. The rotameter was calibrated using a Parkinson-Cowan wet-type gas meter having an accuracy of ± 0.25% at flows

less than 12 sft. 3h<sup>-1</sup>, see Appendix 3. The flow was controlled by a needle valve positioned after the rotameter before passing to the mixing chamber beneath the burners.

#### 7.5 The air supply

Compressed air was available at a pressure of 80-120 lbf.in<sup>-2</sup>g. from a main supply to the laboratory. The passage of methane into the main was prevented by a non-return valve after which the pressure was reduced from about 40 lbf.in<sup>-2</sup>g. by a Norgren model 11400-2G regulator. The air was then passed through an absorber and filter similar to those described in the methane supply. It was found that the absorbents tended to agglomerate and the pressure drop across the absorber increased as progressively more air had been treated until replacement became necessary.

Metric IOP Rotameter with a duralumin float and a mercury manometer.

The pressure at the inlet of the rotameter was controlled at 10 cmHg g. by the regulator. Provision was made for splitting the air flow into two streams. One passed directly to mix with the methane just prior to entry to the mixing chamber, while the second was measured, using a Metric 7P Rotameter with duralumin float and a mercury manometer, before taking up the additive under investigation and then proceeding to the mixing chamber. In practice the former flow path was not used and all the air was utilised in transporting the additive. The air flow was again controlled by a needle valve and copper tubing was used throughout. The rotameters were calibrated with the wet-type gasmeter, Appendix 3.

#### 7.6 The additive supply

Liquid additives were pumped from a 250 ml glass bottle by compressed air at a pressure of 10 cmHg g. The air was controlled by a Norgren model 2A regulator.

The gaseous additives were supplied under pressure in cylinders. The pressure of the gases was reduced to 10 cmHg g. by a Norgren model E2A regulator.

The pressure of the compressed air and thus the liquid additives or the pressure of the gaseous additives were measured with a mercury manometer. Additive flows were metered by Fischer and Porter Flowrators, FP - 1/16 - 10 - G - 5/36 tubes with sapphire floats were used for liquids and with tantulum floats for most of the gases. For hydrogen and helium it was necessary to use an FP - 1/8 - 16 - G - 5/36 tube with a tantulum float. The rotameters were calibrated for the liquids by collecting the liquid passed over a timed interval, see Appendix 3. Where suitable the rotameters were calibrated for the gases using a soap film meter. For dangerous gases a theoretical calibration was obtained using the Fischer and Porter Handbook which incorporates allowances for fluid density and viscosity.

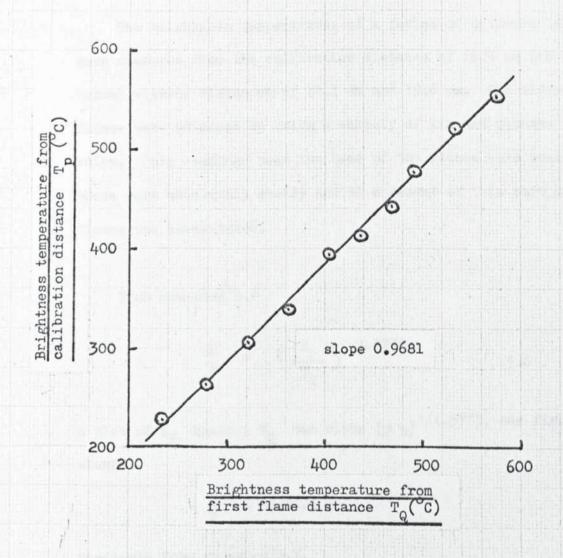
The additive flow was controlled by two needle valves. The first, a combined stopcock and needle valve manufactured from glass and P.T.F.E. was used to restrict the maximum additive flow to full scale on the rotameter. The second needle valve, in stainless steel, was fitted with a needle having a 1° taper and was used for fine control of the additive flow.

After the needle valves the liquid additives were vaporised in glass tubing heated externally by electrical heating tape. The heating rate could be controlled by a variable transformer and provision was made for measuring the voltage and current supplied. The outer surface temperature of the glass tubing was measured approximately by a chromel-alumel thermocouple. The output was measured by a 0 - 10 mV galvanometer recalibrated in C according to B.S.1827.

The vaporised liquid additive or the gaseous additive was drawn into the suction tube of a glass ejector pump, see fig. 20, by the air passing into the mixing chamber. For liquid additives the air stream was heated by external tapes before the ejector, all subsequent tubing and the mixing chamber were similarly heated. As the preliminary tests detailed in section 8.3 show this heating did not itself significantly effect the radiative properties of the flames. Glass tubing was used throughout the additive supply. The purities of the additives used are listed in Appendix 4.

Figure 26.

Plot of brightness temperature measured
at calibration distance against brightness temperature measured at first flame
distance for identical sources



# 8. Preliminary Experiments

# 8.1 Corrections for the portions of the flames viewed and for atmospheric absorption

As has been mentioned in section 5.2.1., the pyrometer views slightly different portions of the two flames. The possibility of atmospheric absorption of radiation by water vapour was discussed in section 5.2.2. A simple preliminary experiment was performed to evaluate the ratios p/q and q/r which were used to correct for these sources of error.

The brightness temperatures of a series of different flames were measured from the calibration distance of 20.3 cm and the normal viewing distances of 14.1 cm and 19.6 cm. The different flames were obtained by using a variety of air and methane flow rates. Only readings near the base of the flames were used as these were absolutely steady and no movement of this part of the flames was encountered.

From equation 5.6

$$\frac{p}{q} = \left(\frac{T_p}{T_Q}\right)^{4.5773}$$

a plot of  $T_p$  against  $T_Q$  had slope  $(p q)^{1/4.5773}$ , see fig. 26, whence

$$\frac{p}{q} = 0.8621$$

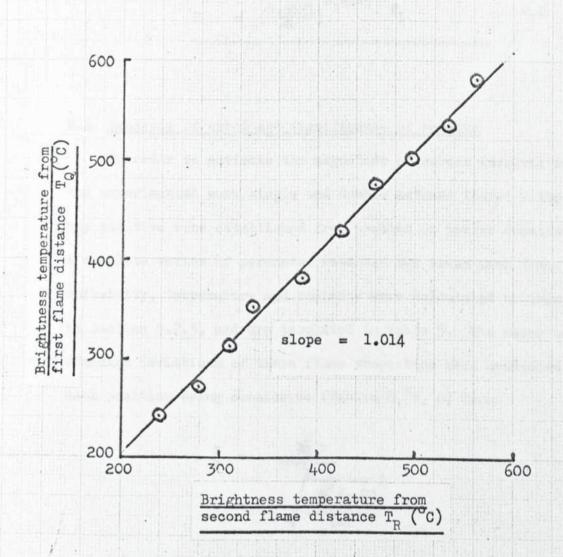
Similarly from equation 5.7,

end by a

$$\frac{q}{r} = (\frac{T_Q}{T_P})^{4.5773}$$

Figure 27.

Plot of brightness temperature measured
at first flame distance against brightness temperature measured at second
flame distance for identical
sources



a plot of  $T_Q$  against  $T_R$  had slope  $(q/r)^{1/4.5773}$ , see fig. 27, whence

$$\frac{q}{r} = 1.067$$

The least mean squares curve fit, Appendix 2, was used to obtain the accurate values of the slopes for substitution in equations 5.12 and 5.13:

$$= 2.067 - 1.067 \left(\frac{T_2}{T_1}\right)$$
 8.1
$$T = \left(\frac{0.8621}{\epsilon}\right)^{0.2185} \cdot T_1$$
 8.2

#### 8.2 Analysis of error and significance of results

In order to estimate the magnitude of errors involved in the experimental work, single and double methane flames without any additive were established from scratch on twelve occasions. A complete series of pyrometer readings was taken each time. The emissivity, temperature and radiance were calculated as described in Section 9.2.3. and are tabulated in Table 5. The means and standard deviations of these flame properties were evaluated for each position being considered (Tables 6, 7, 8) from

$$x = \frac{1}{\sqrt{\frac{x}{N-1}}}$$

$$x = \sqrt{\frac{x}{N-1}}$$

Whence the standard errors of the means were calculated from

4、3分)

Figure 28.

Significance levels and confidence limits of means of emissivities.

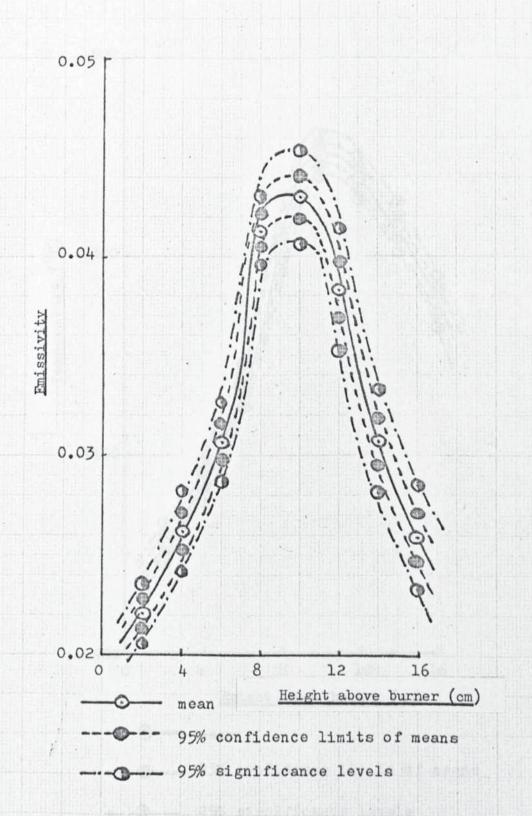


Figure 29.

Significance levels and confidence
limits of means of temperatures.

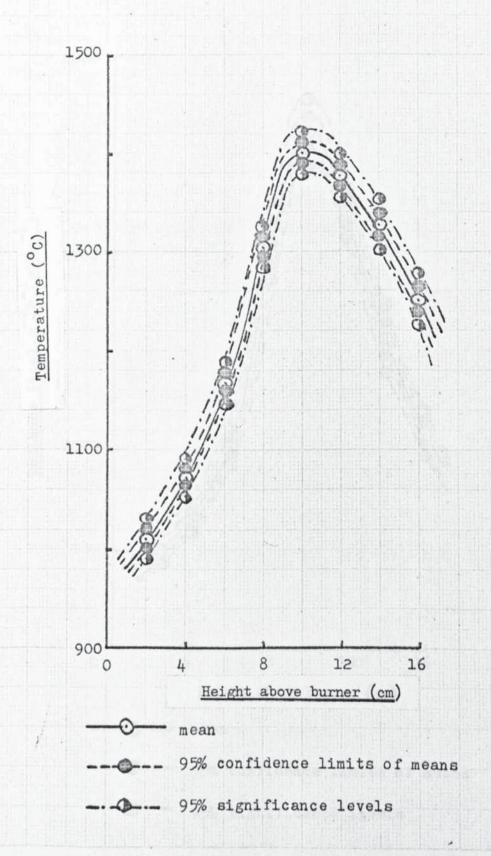
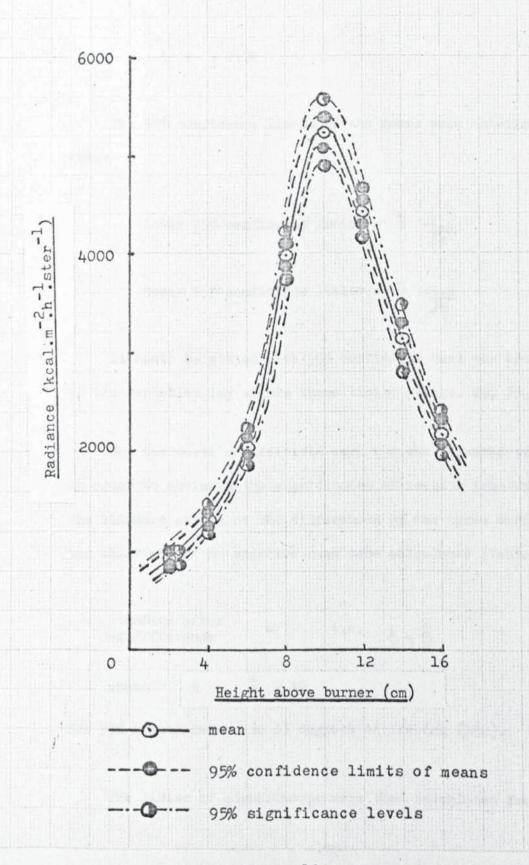


Figure 30.

Significance levels and confidence limits of means of radiances.



standard error of mean = 
$$\frac{t.s}{\sqrt{N}}$$

where from the literature (132) for a probability of 0.05 and 11 degrees of freedom

$$t = \pm 2.20$$

The 95% confidence limits of the means were obtained from:

lower 95% confidence limits = 
$$\bar{x}$$
 -t.s.

upper 95% confidence limits = 
$$\bar{x}$$
 +t.s

It could be stated with 95% confidence that the true values of the variables lay within these limits (figs. 28, 29, 30).

For the tests on additives each run was repeated twice.

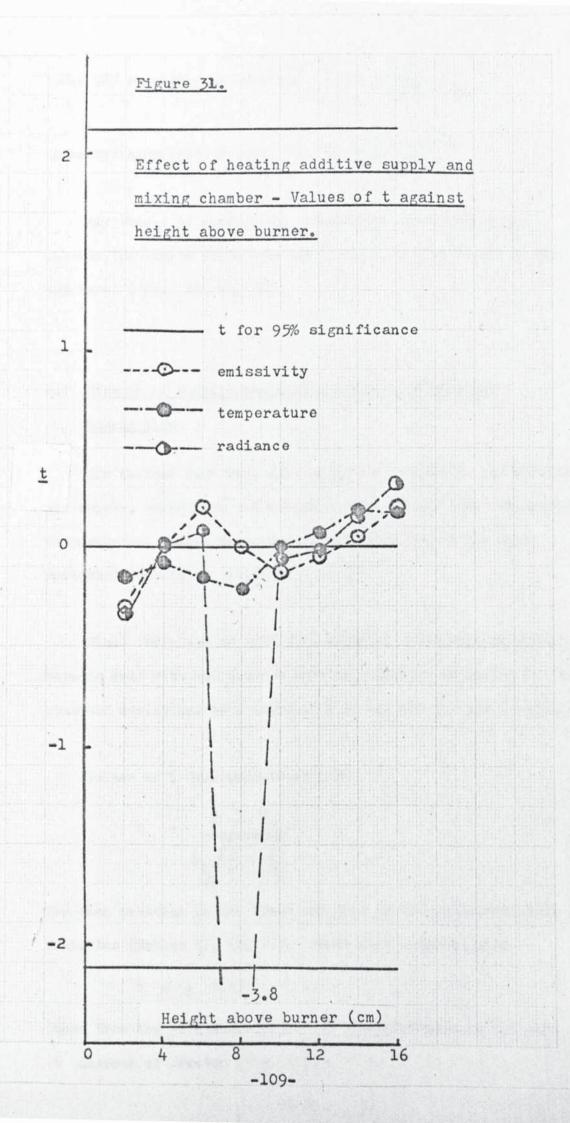
In order to estimate the significance of results from these runs, the standard errors of the differences of the three variables for the additive and no-additive runs were calculated (Table 9) from:

standard error = t.s. 
$$\frac{1}{N} + \frac{1}{N}$$

where 
$$t = \pm 2.16$$

for 95% significance and 13 degress of freedom (132).

The limits of significance were then calculated from:



lower 95% significance level = 
$$\bar{x} - t.s. \sqrt{\frac{1}{N} + \frac{1}{N}}$$

upper 95% significance level = 
$$\bar{x} + t.s. \sqrt{\frac{1}{N} + \frac{1}{N'}}$$

Any values of emissivity, temperature orradiance lying outside the limits calculated above would be significant at the 95% level (figs. 28, 29, 30).

# 8.3 Effects of heating the mixing chamber and additive supply line

Six further runs were made using the heating to the additive air supply, varporiser and mixing chamber (Table 10). Emissivities, temperatures, radiances, means and standard deviations were evaluated (Tables 11, 12, 13).

Since there was no reason to expect a difference in variance between runs with and without heating, combined estimates of the standard deviations were calculated to achieve greater accuracy.

Values of t were calculated from

$$t = \frac{\bar{x} - \bar{x}_{H}}{s_{H}}$$

for each position in the flame and each of the three dependant variables (Tables 11, 12, 13). These were compared with

$$t = \pm 2.12$$

A 109 1.

taken from the literature (132) for a significance of 95% and 16 degrees of freedom (fig. 31).

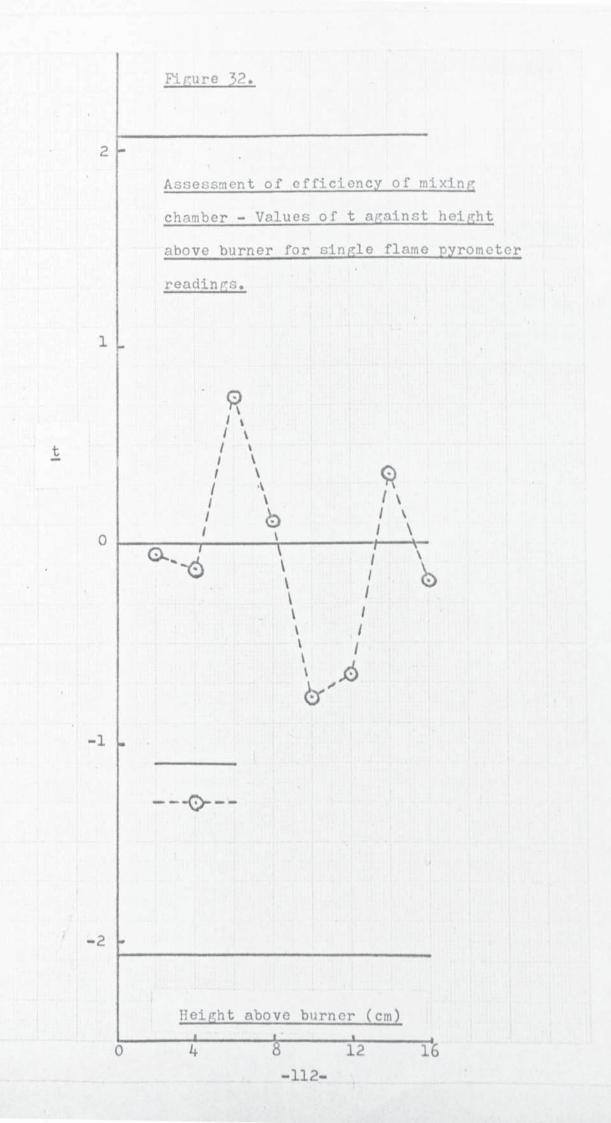
To be significant at the 95% level the experimentally determined value of t would have to be less than -2.12 or greater than +2.12. Since none of the values of t calculated for emissivity and temperature satisfied this condition there was no statistical evidence to suggest that these flame properties were influenced by the heating. In one case, at 8 cm above the burner, the value of t for radiance lay outside the limits of t and the heating was, therefore, statistically significant. Since, however, this was only one of eight and as the variation among the other values appeared to be random, e.g. the values were not consistently positive or negative and did not change regularly along the length of the flame, this was probably a freak result and could be discounted. The heating did not, therefore, have a significant effect on the radiance from the flame.

#### 8.4 Efficiency of the mixing chamber

In order to assess the efficiency of the mixing chamber in supplying similar combustion mixtures to each of the two burners a preliminary experiment was carried out measuring single flame outputs only. The normal flames were set up together with the addition of 20% acetone. Pyrometer readings were taken for each flame at the near flame distance of 14.1 cm with radiation from the other flame being screened from the pyrometer. The experiment was repeated twelve times (Table 14). The means, standard deviations and differences of means for the two burners were evaluated (Table 15).

Values of t were calculated from

$$t = \frac{\bar{x}_1 - \bar{x}_2}{\sqrt[8]{\frac{1}{N_1} + \frac{1}{N_2}}}$$



for each position in the flame (Table 15). These were compared with

$$t = \pm 2.07$$

for a significance of 95% and 22 degrees of freedom (fig. 32).

None of the experimental values of t were less than - 2.07 or greater than + 2.07 so there was no statistical evidence to suggest that the mixing chamber was significantly inefficient and that the two flames were not similar.

#### 9. Procedure and calculation of results

#### 9.1 Experimental procedure

At the beginning of each run the ambient temperature was measured and the calorimeter water temperature indicator was adjusted to this value. The zeroes of all the meters were checked, the pyrometer was levelled and burners were adjusted to be vertical.

The water to the calorimeter was turned on and adjusted to the required flow rate. A single flame was lit and the calorimeter was allowed to heat up until a constant outlet temperature was achieved, air was bled from the calorimeter by way of the vent on the top edge.

When the calorimeter had reached equilibrium, usually after about 30 minutes, the pyrometer outputs for single and double flames without additives were checked. If there were any discrepancies with previous values they were corrected by checking the alignment and level of the pyrometer and burners.

The additive under test was then supplied to the single flame, heating being switched on in the case of liquid additives, and several minutes allowed for the rig to settle down.

The lowest additive flow was tested first. The methane and additive flows were maintained constant while the air flow was adjusted to determine the critical flow for the smoke point, the luminosity just appearing and just disappearing readings being taken consecutively.

The single flame pyrometer output readings were taken next, one reading at flame height increments of 2 cm. These groups of readings were repeated twice and the mean values for the outputs at each height were calculated. The critical air flows for the smoke point were again determined before each group of pyrometer output readings were taken and mean values calculated. After all measurements had been made three times the calorimeter inlet and outlet temperatures were noted.

The additive flow rate was adjusted to the next value and the smoke points and output readings were measured three times. After all the additive rates had been tested on the single flame the second flame was lit and the calorimeter was again allowed to reach equilibrium.

Starting with the lowest additive flow rate the output readings were again taken three times but no smoke point determinations were made with the double flames. Calorimeter inlet and outlet temperatures were again taken after each group of output readings.

Whilst taking pyrometer output readings it was essential to ensure that the flames were stationary and not being distorted by draughts. Provided great care was taken it was possible to obtain consistent results. A small amount of trouble was experienced with the calorimeter due to air bubbles in the water supply. These were bled from the system at the calorimeter itself and the flow rate reset, this tended to drift slightly and required attention about every ten minutes.

The additive metering arrangements proved entirely satisfactory with gases but the rotameter float was prone to pulsation when used with liquids. Adjustment of the additive supply pressure reduced this to a minimum and as it also occurred during calibration of the rotameter it was considered tolerable.

After about twentyfive runs had been performed it was found that the coating of lampblack-water glass solution on the inner surface of the calorimeter had begun to blister. This was most severe where the coating had been applied thickly. It was necessary to scrape off the charred remains and apply a new coating.

The pressure drop through the carbon dioxide-moisture absorber in the air supply had increased to approximately 30 lbf.in<sup>-2</sup> after forty runs when it was found that the carbosorb particles had become welded into a solid mass. Both beds were replaced, this difficulty was not encountered with the absorber in the methane supply however.

#### 9.2 Calculation of results

#### 9.2.1 Smoke point

The stoichiometric air flow was calculated from consideration of the following equation for the complete combination of methane, no account was taken of air required by the additive.

$$CH_4 + 20_2 + 7.52N_2 - C0_2 + 2H_20 + 7.52N_2$$
1 vol. 9.52 vol.

i.e. for a methane flow of  $0.0264 \, 1.5^{-1}$ stoichiometric air flow =  $0.2515 \, 1.5^{-1}$  The actual air flow used was calculated from the air rotameter reading using the equation which had been fitted to the rotameter calibration curve, Appendix 3.

Fraction of Stoichiometric Air = Actual air flow at smoke point 0.2515

# 9.2.2 Calorimeter

The heat transferred to the calorimeter was calculated quite simply from the water flow rate and the temperature difference between the calorimeter inlet and outlet temperatures.

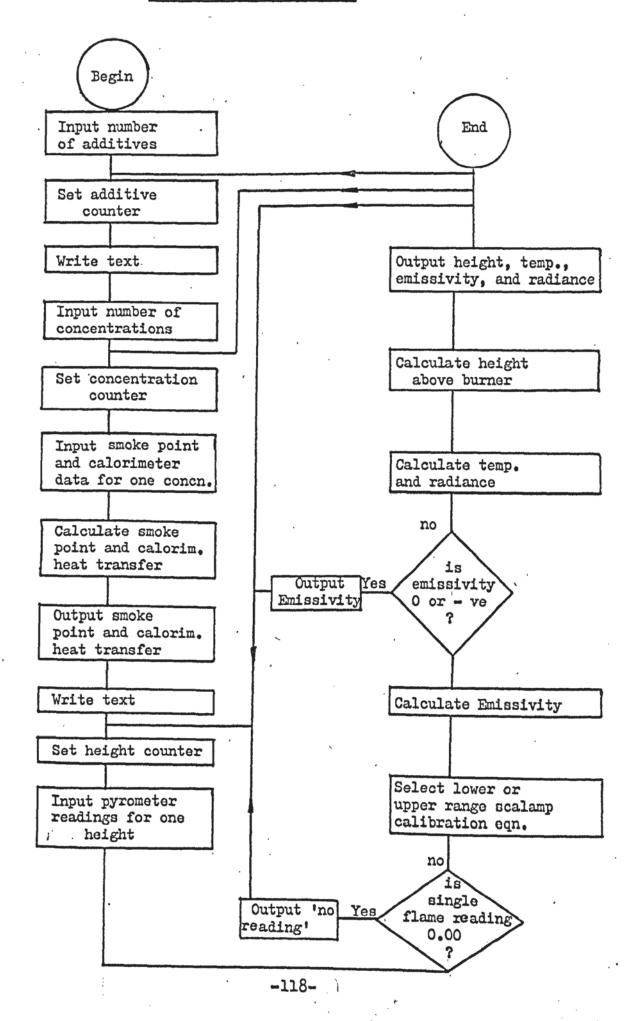
A fixed water flow rate at rotameter reading 18, corresponding to 1.916 ml.s<sup>-1</sup>, was used.

heat transferred to = 1.916 (outlet temp.-inlet temp) cal.s

# 9.2.3 Emissivity, temperature and radiance

Mean values of the three readings of pyrometer output for each pyrometer position were evaluated during the run and entered on the log sheet. The first stage of the calculation of the emissivity, temperature and radiance was the evaluation of the two brightness temperatures using the equations of the galvanometer calibration curves, Appendix 3.

Figure 33. Flowsheet of computer program for calculating final results



lower range:

upper range:

The emissivity, temperature and radiance were then calculated from equations 8.1, 8.2 and 5.14,

$$\epsilon = 2.067 - 1.067 \left(\frac{T_2}{T_1}\right)^{4.5773}$$
8.1

$$T = (\frac{0.8621}{\epsilon})^{0.2185} T_1$$
 °K 8.2

$$W = 1.552.10^{-8}.\epsilon. T^{4} kcal.m^{-2}.h^{-1}.ster^{-1}$$

#### 9.2.4. Computer Programme

In view of the large number of results to be handled the following computer program was written to perform the above calculations. It is summarised by the flowsheet shown in fig.33.

BEGIN' 'COMMENT' THE INFLUENCE OF ADDITIVES ON THE
RADIATIVE PROPERTIES OF A PREMIXED
METHANE AIR FLAME:
ROTAM, SINLETT, SOUTLETT, DINLETT, DOUTLETT,
AIRFLO, FRASTO, SHEATTR, DHEATTR, RDG 1,
RDG 2, CT1, T1, CT2, T2, EMISSIVITY, CT,
RADIANCE;

```
'INTEGER' NOADDS, NOCONCNS, HEIGHT, M. N. J:
'REAL' 'ARRAY' CONCN 1:5 ;
CONCN 1 : = 2.5;
CONCN 2 : = 5.0;
CONCN 3 := 10.0;
CONCN 4 := 15.0;
CONCN 5 := 20.0;
NOADDS : = READ:
'FOR' M := 1 'STEP' 1 'UNTIL' NOADDS 'DO'
          WRITE TEXT ('("('3C')'RUN'('S')'NO'('S')")');
          COPYTEXT ('('ZZ')'):
          WRITE TEXT ('(''('20S')'ADDITIVE'('25')'')');
          COPYTEXT ('('ZZ')');
          NOCONCNS := READ;
           'FOR' N := 1 'STEP' 1 'UNTIL' NOCONCNS 'DO'
          ROTAM := READ;
'BEGIN'
          SINLETT := READ:
          SOUTLETT := READ:
          DINLETT := READ:
          DOUTLETT := READ;
          AIRFLO := 0.01271 + 0.01175*ROTAM + 0.0000954*
                       ROTAM ***12:
          FRASTO := AIRFLO/0.2515;
          SHEATTR := 1.916* (SOUTLETT-SINLETT);
          DHEATTR := 1.916* (DOUTLETT-DINLETT);
          NEWLINE (4);
          SPACE (9);
          PRINT(CONCN N ,2,1);
WRITE TEXT ('('%'('S')'CONCENTRATION'('3C,16S
           ')'SMOKE'('S')'POINT'('(2C,22S')'MEAN'('S')'AIR'('S')'
          ROTAMETER '('S')'READING'('16S')'')');
          PRINT(ROTAM, 2, 1);
          WRITE TEXT ('(''('C,22S')'FRACTION'('S')'OF'('S')'
STOICHIOMETRIC '('S')'AIR'('S')'FLOW'('8S')'')');
          PRINT (FRASTO, 1,3);
          WRITE TEXT ('(''('2C,16S')'CALORIMETER'('2C,40S')'
          INLET '('S')'TEMP'('55')'OUTLET'('S')'TEMP'('6S')'HEAT
          '('S')'TRANSFER'('C,42S')'(DEG.C)'('7S')'(DEG.C)'
           ('10S')'(CAL/S)'('C,22S')'SINGLE'('S')'FLAME'('9S')
          11)1);
          PRINT (SINLETT, 2,1);
          SPACE (7);
          PRINT (SOUTLETT, 2,1);
          SPACE (9);
          PRINT (SHEATTR,2,1);
WRITE TEXT ('(''('C,22S')'DOUBLE'('S')'
          FLAME'('9S')'');
          PRINT (DINLETT, 2, 1);
          SPACE (7);
          PRINT (DOUTLETT, 2, 1);
          SPACE (9);
          PRINT (DHEATTR, 2,1);
          WRITE TEXT ('('3C, 16S')'EMISSIVITY'('S')'
          TEMPERATURE ('S') RADIANCE ('2C, 22S')
          HEIGHT'('S')'ABOVE'('SS')'PYROMETER'('S')'
          READINGS'('3S')'BRIGHTNESS'('S')'TEMP'('
          3S')'EMISSIVITY'('3S')'TEMPERATURE'('5S')'
```

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```
RADIANCE'('C.22S')'BURNER'('S')'(CM)'('4S')'
                  SINGLE'('6S')'DOUBLE'('3S')'SINGLE'('3S')'
                  DOUBLE'('18S')'(DEG.C)'('5S')'(KCAL/SQ.M.STER)
                  1(101)11)1);
                  HEIGHT := 0:
                  'FOR' J := 1 'STEP' 1 'UNTIL' 8 'DO'
        *BEGIN*
                  RDG1 := READ;
                  RDG2 := READ:
                  'IF' REG1 'EQ' 0.00 'THEN' 'GOTO' NORDG:
                  'IF' RDG2 'GE' 20.00 'THEN' 'GOTO' RANGE 2;
        RANGE 1: CTl := 180.31+41.486*RDG1 - 1.3024*RDG1****2;
                  'GOTO' CONT 1;
                  RDG1 := RDG1 - 20.00;
        RANGE 2:
                  CT1 := 384.12 + 111.37*RDG1 - 3.8219*RDG1****2;
        CONT1:
                  T1 := CT1 + 273.16;
                  'IF' RDG2 'GE' 20.00' THEN' 'GOTO' RANGE 4:
                  CT2 := 180.31+41.486*RDG2-1.3024*RDG2****2:
        RANGE 3:
                  'GOTO' CONT 2;
        RANGE 4:
                  RDG2 := RDG2-20.00;
                  CT2 := 384.12+111.37*RDG1-3.8219*RDG2****2:
        CONT2:
                  T2 = CT2 + 273.16;
                  EMISSIVITY := 2.067-1.067*(T2/T1)!**!4.5773;
                  'IF'EMISSIVITY'LE'O.000'THEN'GOTOFAILURE;
                  CT := T1*(0.8621/EMISSIVITY)'**'0.2185-273.16:
                  RADIANCE := 1.552*10****(-8)*EMISSIVITY*
                                        (CT+273.16) *** 4:
                  HEIGHT := HEIGHT + 1:
                  SPACE (26);
                  PRINT (HEIGHT, 2,0);
                  SPACE (6);
                  PRINT (RDG1,2,2);
                  SPACE (4);
                  PRINT (RDG2,2,2);
                  SPACE (1);
                  PRINT (CT1,3,1);
                  SPACE (2);
                  PRINT (EMISSIVITY, 1, 3);
                  SPACE (6);
                  PRINT (CT,4,0);
                  SPACE (7);
                  PRINT (RADIANCE.5.1):
                  'GOTO' AGAIN;
                  WRITE TEXT ('(''('10S')' NO'('S')'READING ')');
        NORDG:
                  'GOTO' AGAIN;
                  WRITE TEXT ('(''('10S')'NEGATIVE'('S')'
        FAILURE:
                           EMISSIVITY'('4S')'');
                  PRINT (EMISSIVITY, 1, 3);
        AGAIN:
                  NEWLINE (1):
        END:
        END:
        'END';
*END*:
```

The programe was run on an ICL 1905 computer.

### 10. Discussion of results

The full results from the research are listed in

Appendices 6 and 7. To facilitate comparisons between

the effects of the different additives they have been

classified into eight groups. These are paraffins;

aromatics; acetylene, ethylene and cyclohexane;

halogenated compounds; organic compounds containing oxygen;

flame reactants and products of combustion; miscellaneous

additives, and, finally, sulphur trioxide.

#### 10.1 Calorimeter

The results obtained for heat transferred to the calorimeter were largely insignificant (Appendix 5).

Those additives which could be considered as fuels
usually gave increased heat transfer rates as the additive
flow was increased but the effects observed within groups
of similar additives were random. It has not been possible
to correlate the heat transfer rates with either emissivity,
flame temperature or radiance. The influence of these factors
on the heat transfer was so slight that changing conditions
within the laboratory such as room temperature and the presence
of draughts rendered them insignificant. It was not considered
worthwhile expending further effort on improving the rig as the
data of this type available on such a small scale would be of
rather limited value. It was decided to concentrate on
measuring the other properties of the flame.

Significant overall heat transfer could only be effectively measured in a model furnace using water-cooled loads on a scale many times larger e.g. as in the experimental furnaces at Ijmuiden. This would be facilitated by the bulk supplies of natural gas now more readily available.

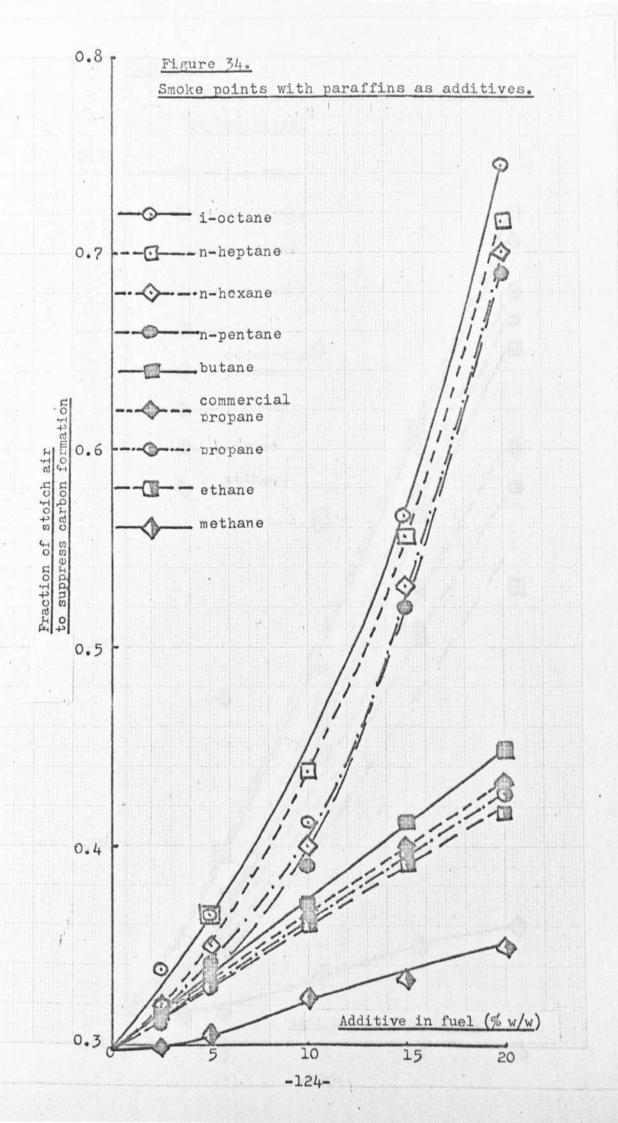
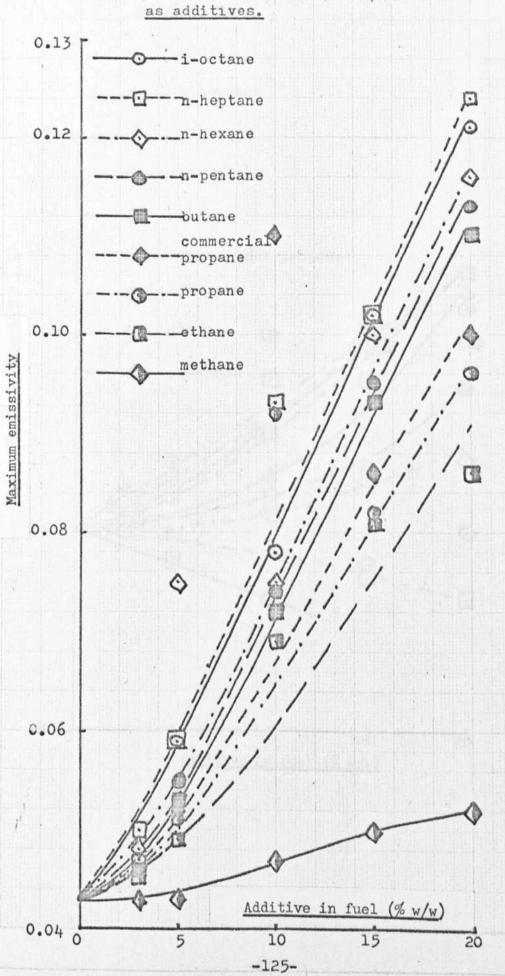


Figure 35.

Maximum emissivities with paraffins



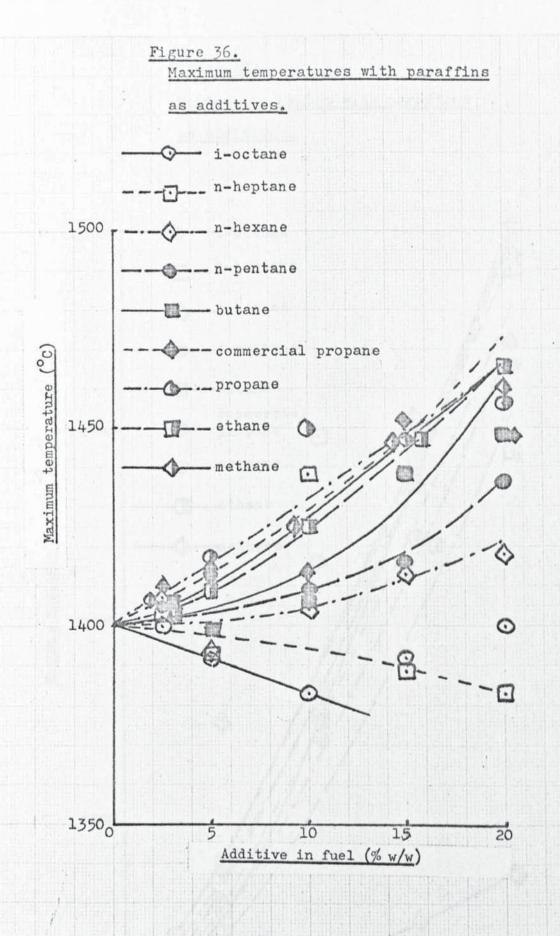
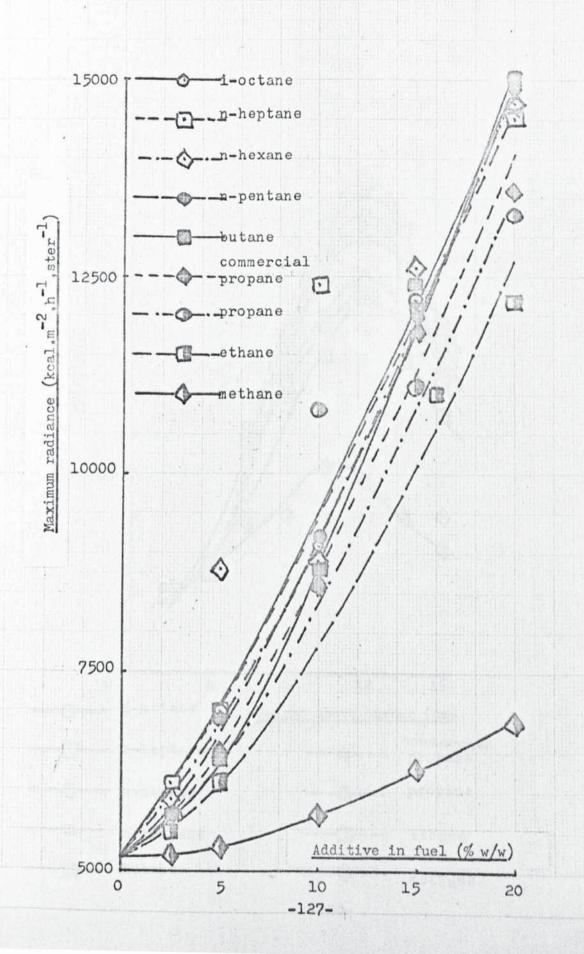


Figure 37.

Maximum radiances with paraffins as additives.



Variation of emissivity with height above burner for 20% w/w paraffin additives.

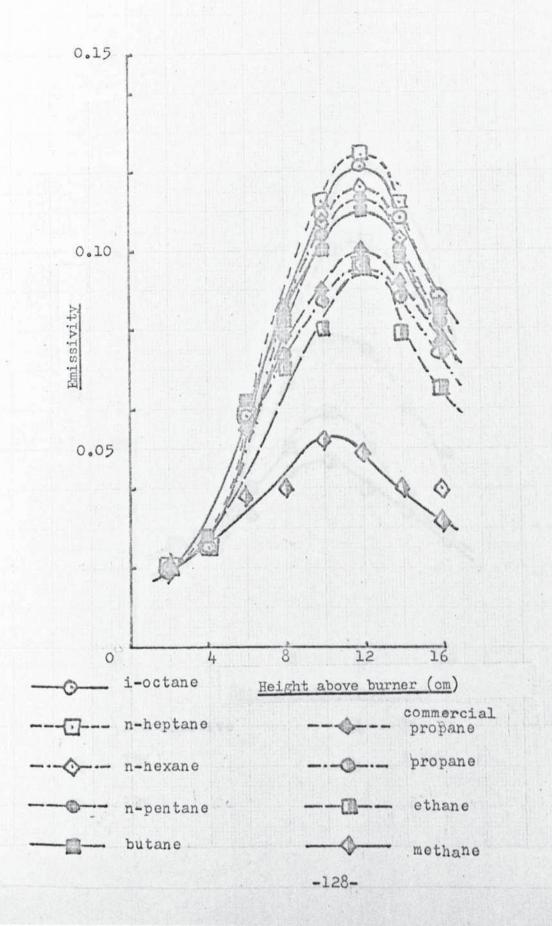
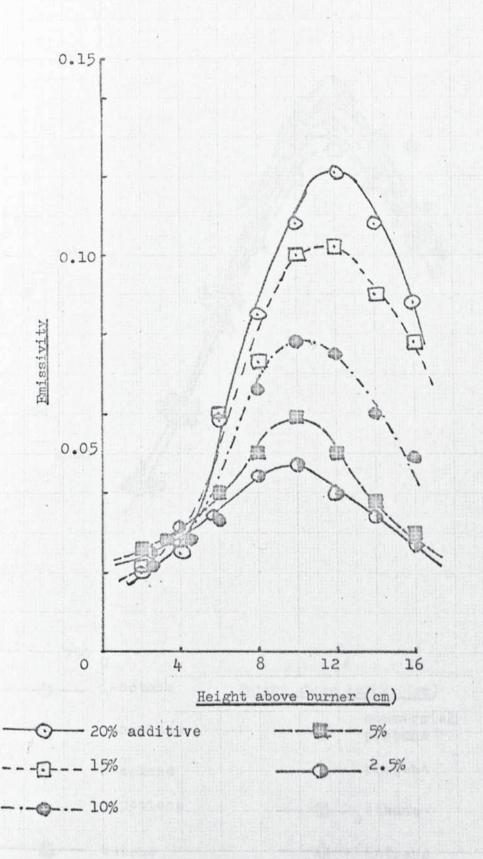
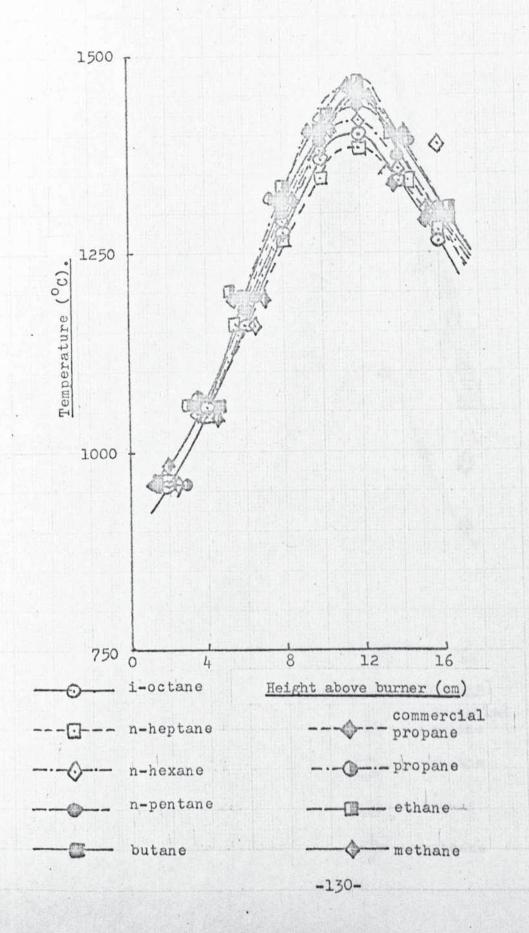


Figure 39.

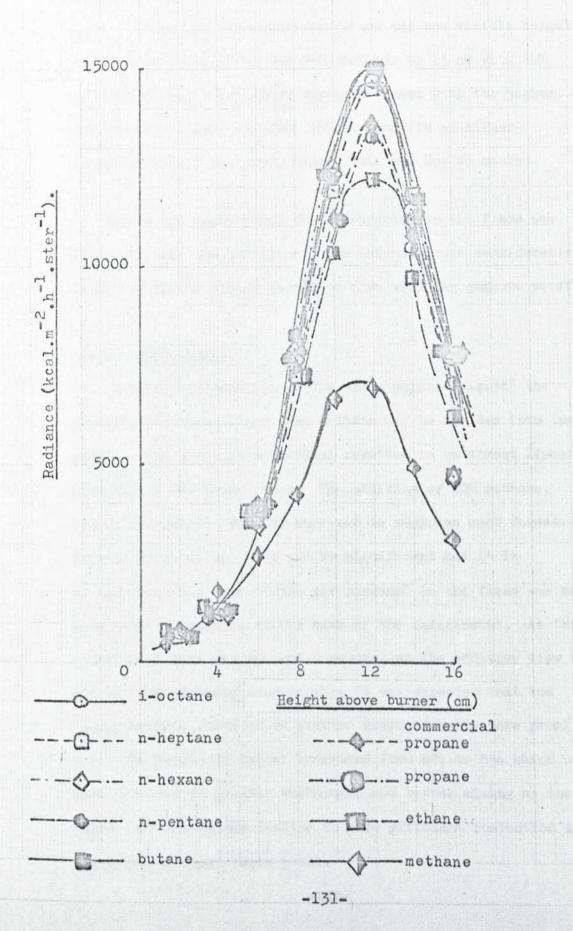
Variation of emissivity with height above burner for i-octane as additive.



Variation of temperature with height above burner for 20% w/w paraffin additives.



Variation of radiance with height above burner for 20% w/w paraffin additives.



## 10.2 Paraffins as additives

appearance. The main body of the flame was deep blue with a pale blue inner cone of height approximately 6 cm. The maximum width was approximately 2.5 cm and the visible height ranged from about 10 cm for pure methane to 15 cm with 20% i-octane added. The flames appeared longer with the higher additive flow rates and also when a paraffin of higher molecular weight was used, though this was not so marked.

Above the smoke point the colouration in the flame was yellow for all the paraffins. The intensity was considerably greater with the liquid paraffins than with the gaseous paraffins.

## 10.2.1 Smoke Point

From a consideration of the smoke points (fig.34) the paraffin additives other than methane may be divided into two groups. The addition of methane resulted in an almost linear increase in the smoke point. The addition of 20% methane, however, required only 17% more air to suppress soot formation. This discrepancy may well not be significant but it is surprising since some of the air consumed in the flame was not premixed but drawn in at the base of the calorimeter. As the quantity of this air was not dependant on the additive flow rate and did not increase accordingly, it was expected that the higher methane rates would require proportionally more premixed air. The Reynold's Number increased from 642 to 856 which would have resulted in greater turbulence and better mixing as the gases left the burner leading to more efficient combustion and, therefore, a lower smoke point.

The gaseous paraffins, ethane, propane, butane and the commercial propane, which contained a high proportion of butane, also caused a linear increase in the smoke point. The commercial propane was tested because it represented a quality of liquified petroleum gas readily available to industry. The differences between the four were quite small, although the smoke point was undoubtedly higher for the higher molecular weights. The differences between these paraffins and methane were much more marked, the smoke point of the latter being considerably lower. This is due to the variety of  $C_1$  and  $C_2$  radicals which can result from the dehydrogenation and breakdown of the higher paraffins, whereas the methane can only be dehydrogenated to  $C_1$  radicals. It is thus possible for the former to build up more quickly into polyacetylenes, polyacetylene radicals and eventually into soot.

The liquid paraffin additives showed quite different effects, proportionally more air being required to prevent soot formation as the concentration of the additive was increased. The four liquids behaved similarly, though the higher molecular weights corresponded to the higher smoke points as with the gaseous paraffins. The differences between the smoke points decreased at concentrations above 10%. The shape of the curves and the difference in behaviour between the liquid and the gaseous paraffins suggest there was some condensation of the liquid additive vapour in the burner tubes. Small droplets would have been formed, carried into the flame and thermally cracked to relatively large soot particles. These would have had a lower

emissivity than a larger number of smaller particles formed by a synthesis mechanism.

# 10.2.2. Emissivity, temperature and radiance

Plots of the maximum emissivity, temperature and radiance against concentration for each additive are shown in figs. 35, 36, 37. All the paraffins caused an increase in the maximum emissivity of the flames. There is some scatter among the points on these curves but above a concentration of 5% they can reasonably be approximated to straight lines. In general the higher molecular weights favoured the higher emissivities, the only exception being i-octane. When studying paraffins as the fuels of pre-mixed flames, Street and Thomas showed that the branched chain isomers had lower smoke points than the straight chain isomers (85). In this research the flames with i-octane added had lower emissivities even than those with n-heptane; the smoke points, however, did not show similar behaviour, the smoke point with i-octane being greater than that with n-heptane. It should be noted that the differences were very small in each case.

The increases in emissivity caused by adding higher paraffins were much greater than those achieved by adding methane itself, presumably for the same reasons as were discussed in the previous section. The differences, however, were even more noticeable than with the smoke points. The soot particles formed by thermal cracking of the liquid in the droplets would have been larger but much fewer in number than those formed by a polymerisation

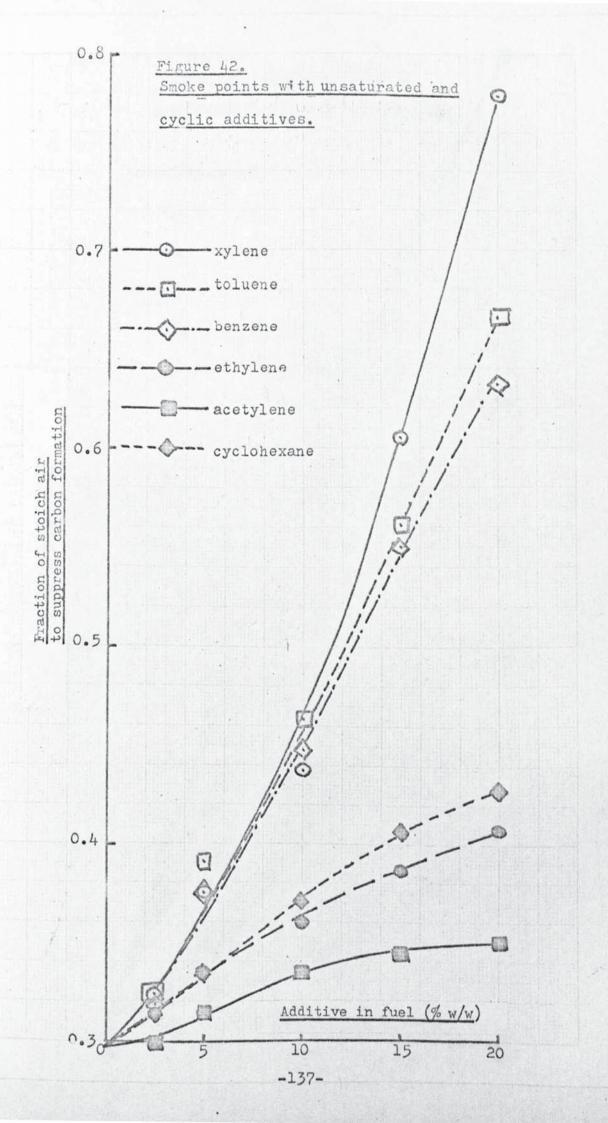
process so that a large increase in emissivity need not necessarily occur. It can be shown that for a cloud of particles with a constant particle mass concentration the emissivity is inversely proportional to the particle diameter (Appendix 1).

The maximum emissivity of the flames could be raised to 0.11 - 0.125 by adding 20% of n-pentane, n-hexane, n-heptane or i-octane, corresponding to flames lm thick with maximum emissivities in the range 0.9 - 0.99. The proportion of the flames at this high emissivity, however, was relatively small (figs. 38, 39). For all the paraffins except methane the maximum emissivity occurred at the same position above the burner; this height was a function of the additive concentration and ranged from about 9 cm at 2.5% concentration to 12 cm at 20% concentration. The shift was gradual as the concentration was increased. The fall off in emissivity on both sides of the maximum emissivity position was rapid and the high level was not sustained along the length of the flame as would be desirable in an industrial furnace. The position of maximum emissivity with methane added to the flame moved only from 9 cm to just over 10 cm above the burner as the concentration was increased.

The flame temperature was also dependent on the molecular weight of the additive, but inversely on this occasion. The i-octane was again an exception, its flame being hotter than that containing n-heptane. The flames with the higher emissivities lost more heat by radiation to the calorimeter than the others and were consequently slightly cooler.

The variation in temperature between the different flames was low. The maximum temperature occurred in almost the same position for all the additives at all concentrations, approximately 12 cm above the burner (fig. 40). This coincided with the position of maximum emissivity when 20% additive concentrations were used and gave rise to very high radiances under these conditions.

As the plot of radiance against height above the burner at 20% additive concentration shows (fig. 41), increases in emissivity were sufficient to offset the losses in temperature for all the additives. This was the case at all concentrations and it can be seen that all the additives gave a considerable improvement over methane, a high molecular weight additive favoured a high radiance. It is interesting to note that butane and i-pentane both yielded a higher radiance than n-hexane, n-heptane and i-octane. Although the latter had higher emissivities than the former, there was so much cooling of the flames that there was a resultant decrease in radiance. This may not occur in larger flames but it is an important factor when considering the use of additives to enhance the radiating properties of flames.



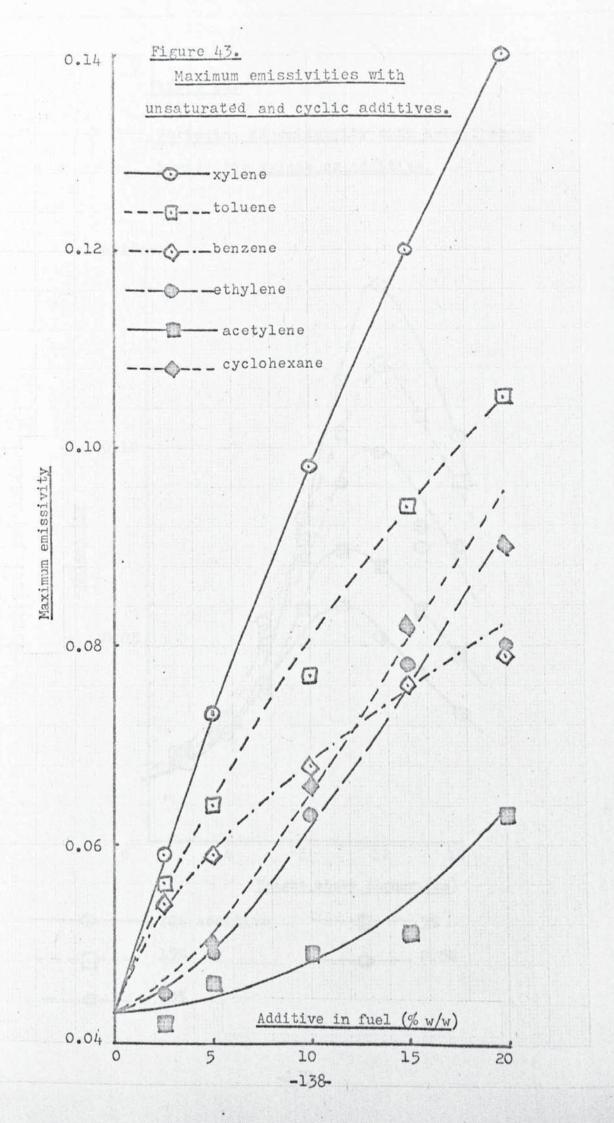


Figure 44.

Variation of emissivity with height above burner for xylene as additive.

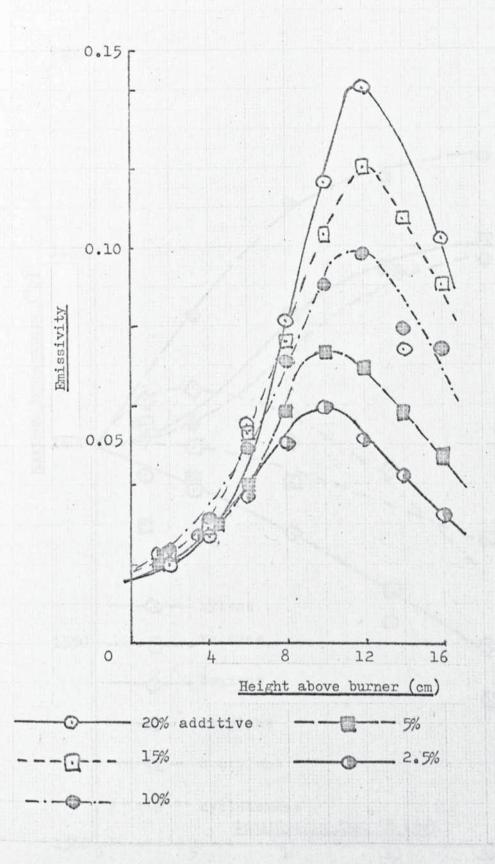
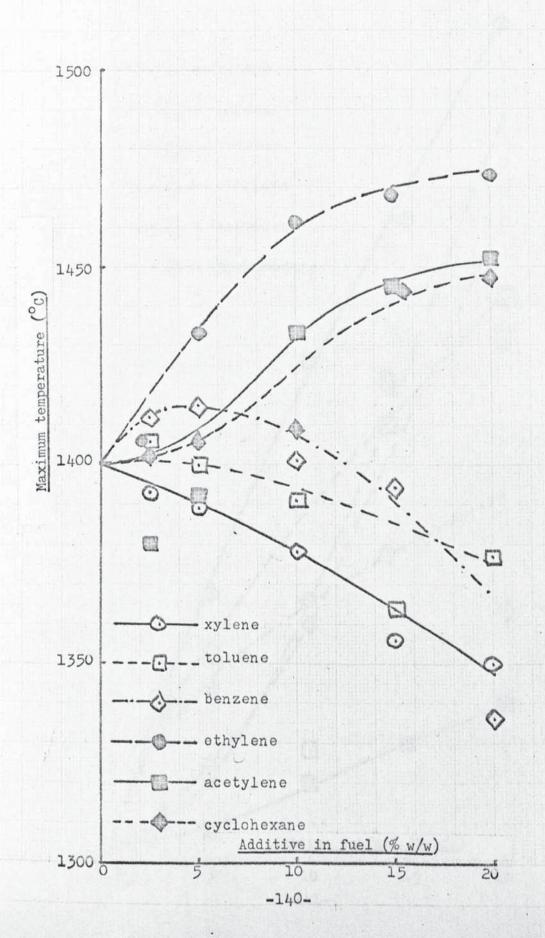
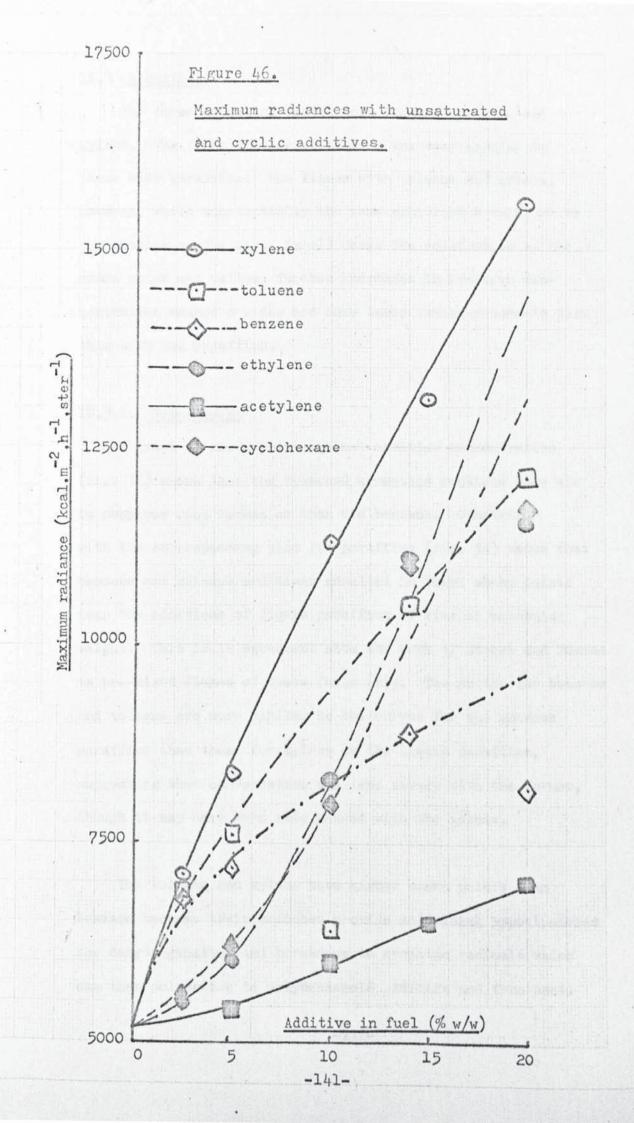


Figure 45.

Maximum temperatures with unsaturated and cyclic additives.





## 10.3 Aromatics

The three aromatics tested were benzene, toluene and xylene. The flame containing benzene was very similar to those with paraffins. The flames with toluene and xylene, however, while substantially the same exhibited a faint white colouration at the tip. In all cases the colouration at the smoke point was yellow, further increases in additive concentration caused a wider and more bushy luminous zone to form than with the paraffins.

## 10.3.1. Smoke point

A graph of smoke point against additive concentration (fig. 42) shows that the branched aromatics required more air to suppress soot formation than did benzene. Comparison with the corresponding plot for paraffins (fig. 34) shows that benzene and toluene additions resulted in lower smoke points than the additions of liquid paraffins of similar molecular weight. This is in agreement with the work of Street and Thomas on pre-mixed flames of these fuels (85). The curves for benzene and toluene are more similar to the curves for the gaseous paraffins than those for xylene or the liquid paraffins, suggesting that condensation was less severe with the former, though it may have been more marked with the xylene.

The toluene and xylene have higher smoke points than benzene because their branches provide additional opportunities for dehydrogenation and breakdown to aromatic radicals which can then polymerise to polybenzenoid radicals and form soot.

# 10.3.2. Emissivity, temperature and radiance

The lower concentrations of the aromatic additives produced much greater increases in maximum emissivity of the flames when compared with the paraffins (figs. 35, 43). The highest emissivity recorded during all the experimental runs was 0.14 for 20% xylene but as with the smoke points the toluene and benzene emissivities at the higher concentrations fell slightly below those of the liquid paraffins. The range of emissivities of the aromatics at any particular concentration was much greater than for the liquid paraffins, exphasising that in respect of soot formation the significant features of their molecular structures are the branches. The branches provide the opportunities for soot formation to occur and the benzene rings provide the bulk of the deposit.

As little as 10% xylene gave a maximum emissivity of 0.098 which corresponds to an emissivity of almost 0.9 on an industrial scale. Unfortunately, as with the paraffins, this high level was not sustained over a significant length of the flame (fig. 44). 20% xylene gave an emissivity of 0.1 over approximately one third of the flame's length which is a marked improvement but a high emissivity over almost the entire flame length is desired. The position of maximum emissivity was dependent on the additive concentration as it was with the paraffins, though the lower regions of the flame, up to 5 cm, showed rather lower emissivities with the aromatics and the fall off at higher levels in the flame was not quite so rapid.

Whereas the addition of up to 10% benzene resulted in an increase in the maximum flame temperature, toluene and xylene cooled the flame at all addition rates. Compared with the liquid paraffins the cooling effect of xylene was even more severe due to the higher emissivities which occurred (fig. 45).

The maximum radiance of the flames with xylene added was greater than with any other additive at any concentration, the high addition rates yielding diminishing benefits, however (fig. 46). Toluene and benzene were more effective than the paraffins for producing a high radiance at low rates of addition but at increased levels their efficiency deteriorated and the radiance was considerably lower than when using liquid paraffins. This was particularly noticeable in the case of benzene.

# 10.4. Acetylene, ethylene, cyclohexane

The flames with acetylene added were rather different from those with other hydrocarbon additives in that at the smoke point, the yellow colouration appeared at the tip of the inner cone rather than at some distance above this point. This is in contrast to 100% acetylene flames in which the luminous zone occurs over the whole of the flame front at a position well above the inner cone, similarly to a 100% methane flame (71, 72, 73). The presence of the additive acetylene and thence radicals such as  $C_2H$ , CH,  $C_2$  at an earlier position than could be achieved in a pure methane flame enables the polymerisation to polyacetylenes and polyacetylene radicals to be initiated and completed near to the reaction zone.

#### 10.4.1 Smoke point

The plot of smoke point against concentration for acetylene is very similar to that for methane (figs. 34, 42). Only slightly more air was necessary with the acetylene and this was much less than with ethane and the other paraffins. Although acetylene promoted soot formation earlier in the flame it did not significantly increase the tendency of the flame to soot.

The ethylene flames were very similar in appearance to those of pure methane and the effect of ethylene on the smoke point was akin to that of ethane (fig. 42). When comparising the flames of 100% ethane and 100% ethylene, Street and Thomas found that ethane caused a more marked increase in the smoke point than ethylene (85).

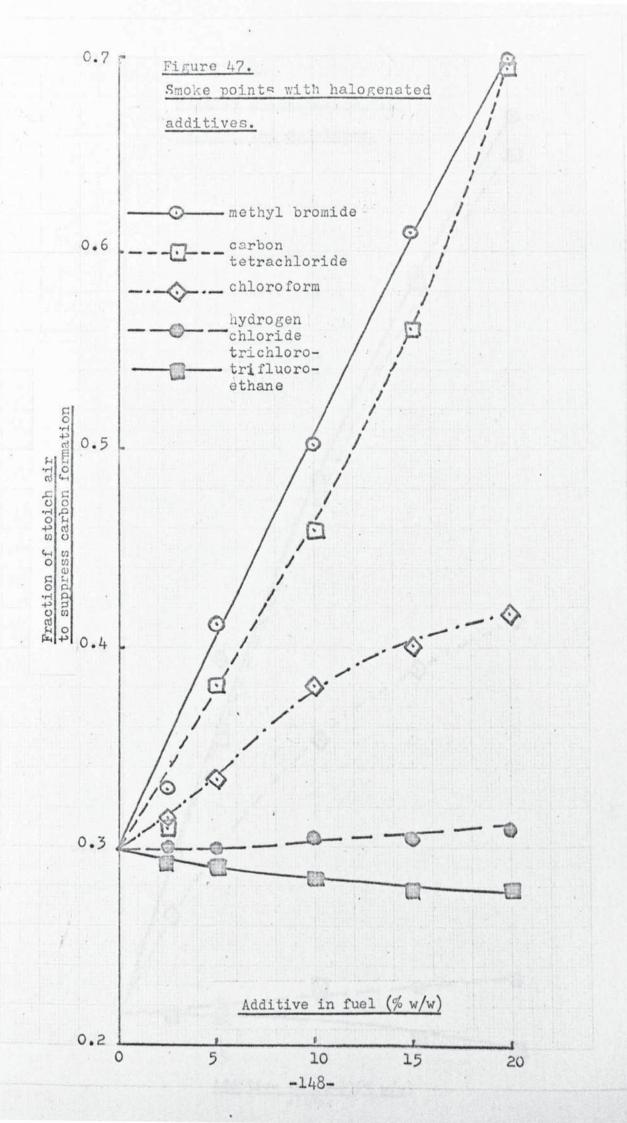
The ring structure of cyclohexane made the formation of radicals more difficult than with the other paraffins of similar molecular weight, and it behaved in the same way as propane. In flames richer than at the smoke point the yellow colouration was generally paler than with other paraffins. At the higher concentrations the flames were noticeably longer.

# 10.4.2 Emissivity, temperature and radiance

Acetylene had little influence on the maximum flame emissivity below 5% concentration (fig. 43). At 10% and above it became more effective than methane itself but was still inferior to ethane. The point of maximum emissivity was slightly nearer to the burner than with the paraffins due to the earlier formation of soot. The shape of the emissivity-concentration curves for ethylene and cyclohexane also followed the pattern of the paraffins rather than the aromatics (figs. 35, 43). The cyclohexane flames had slightly higher emissivities than the ethylene flames but they were both very similar to ethane and propane flames. At 15% concentration and above they had greater emissivities than benzene flames; however, none of these additives gave a worthwhile increase in the flame emissivity.

Even at low concentrations the ethylene flames were much hotter than the acetylene or cyclohexane flames which were more closely related to the paraffin flames although the 20% additions did not achieve quite such high temperatures (fig. 45).

The radiance of the ethylene and cyclohexane flames were similar, due to the relatively low emissivity and high temperature of the former and the higher emissivity and lower temperature of the latter. In both cases at the higher additive concentrations the effect on the radiance was proportionally greater than at low concentrations. The readiances of those flames with acetylene added were only a little more than those with methane (fig. 46).



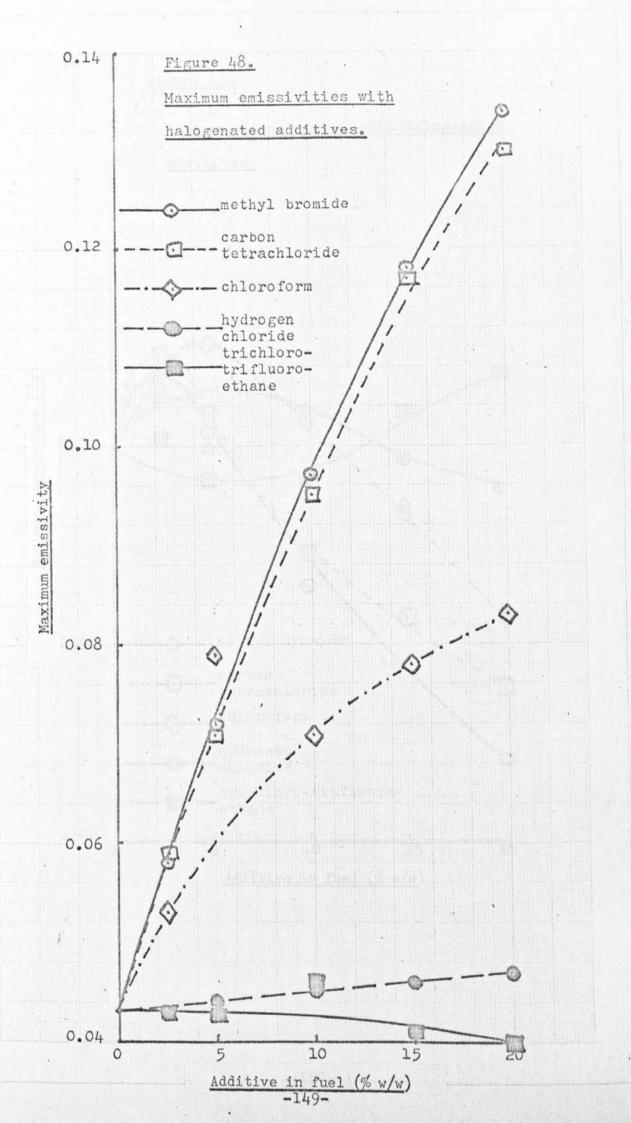


Figure 49.

Maximum temperatures with halogenated additives.

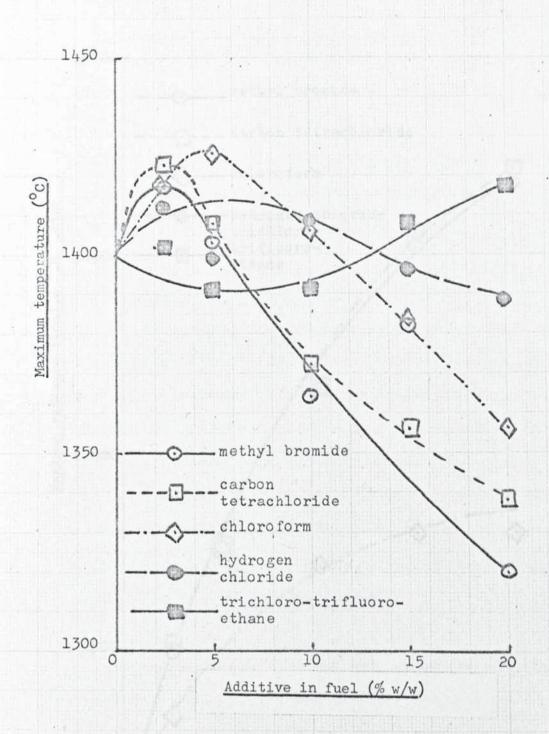
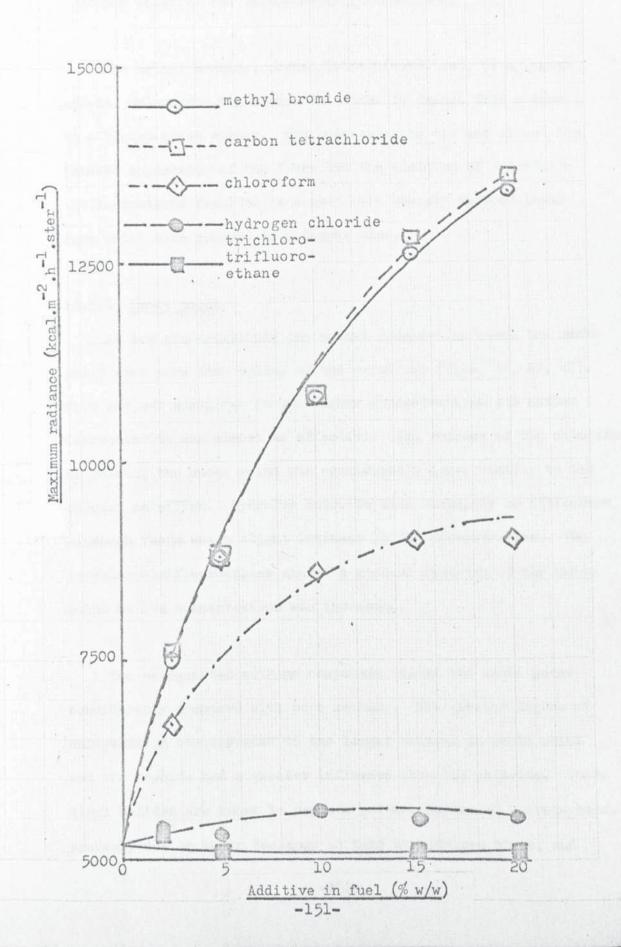


Figure 50.

Maximum radiances with halogenated additives.



# 10.5 Halogenated compounds

The five halogenated compounds tested were methyl bromide and carbon tetrachloride, both known flame retardants, chloroform, hydrogen chloride and trichloro-trifluoroethane.

The methyl bromide, carbon tetrachloride and, to a lesser extent, chloroform all caused the flame to change from a blue to a bluish-green colour. Hydrogen chloride did not affect the general appearance of the flame but the addition of trichlorotrifluoroethane resulted in a much more sharply defined inner cone which also changed to a deeper blue.

#### 10.5.1 Smoke point

At low concentrations the methyl bromide increased the smoke point even more than xylene or the paraffins (figs. 34, 42, 47). This was not sustained at the higher concentrations and carbon tetrachloride was almost as effective. The success of the chloroform in raising the smoke point was considerably less, similar to the gaseous paraffins. Hydrogen chloride made virtually no difference although there was a slight increase at 20% concentration. The trichloro-trifluoroethane caused a gradual lowering of the smoke point as its concentration was increased.

The halogenated methane compounds raised the smoke point considerably compared with pure methane. The greater degree of halogenation corresponded to the larger changes in smoke point and the bromide had a greater influence than the chloride. These alkyl halides are known to promote polymerisation of hydrocarbons, probably due to their tendency to take up hydrogen atoms, and

when used as an additive in this way they are able to increase the rate of polymerisation of the acetylene radicals.

# 10.5.2 Emissivity, temperature and radiance

The plots of maximum emissivity against concentration for methyl bromide and carbon tetrachloride are very similar to that for xylene (figs. 43, 48). The methyl bromide has a slightly greater influence than the carbon tetrachloride and the xylene is a little more effective than both. As in the case of xylene the maximum emissivity which can be achieved with as little as 10% addition is acceptably high at 0.096, corresponding to almost 0.9 on an industrial scale, but again this level is maintained along only a small proportion of the flame. The fall off in emissivity on both sides of the maximum is quite rapid.

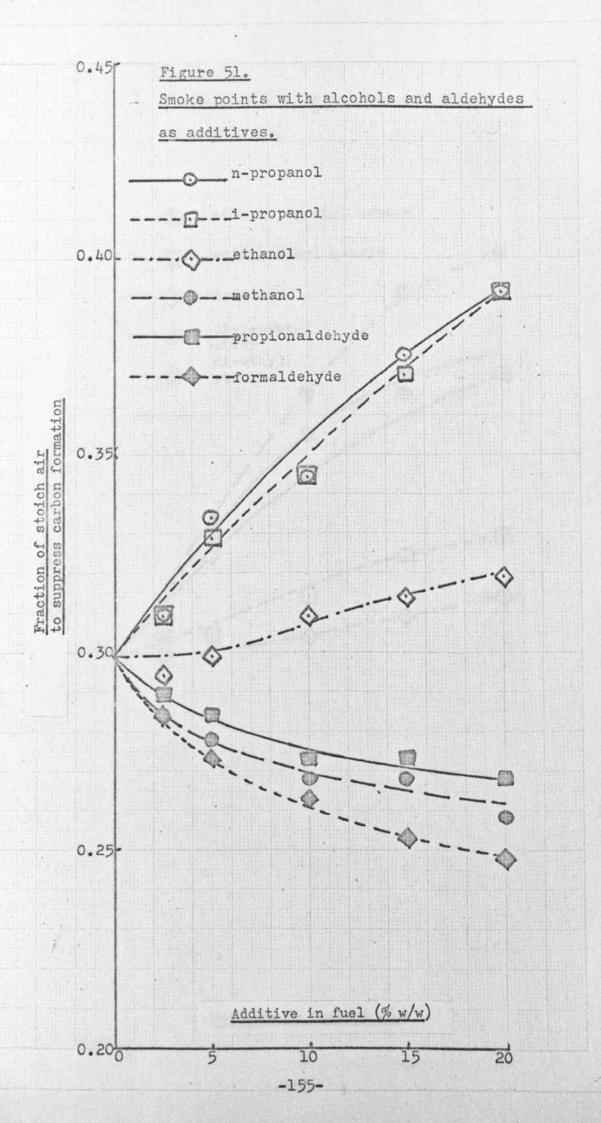
The carbon tetrachloride is less effective than the other alkyl halides, the shape of the emissivity-concentration curve and the maximum emissivity of 0.083 at 20% concentration are very similar to those for benzene.

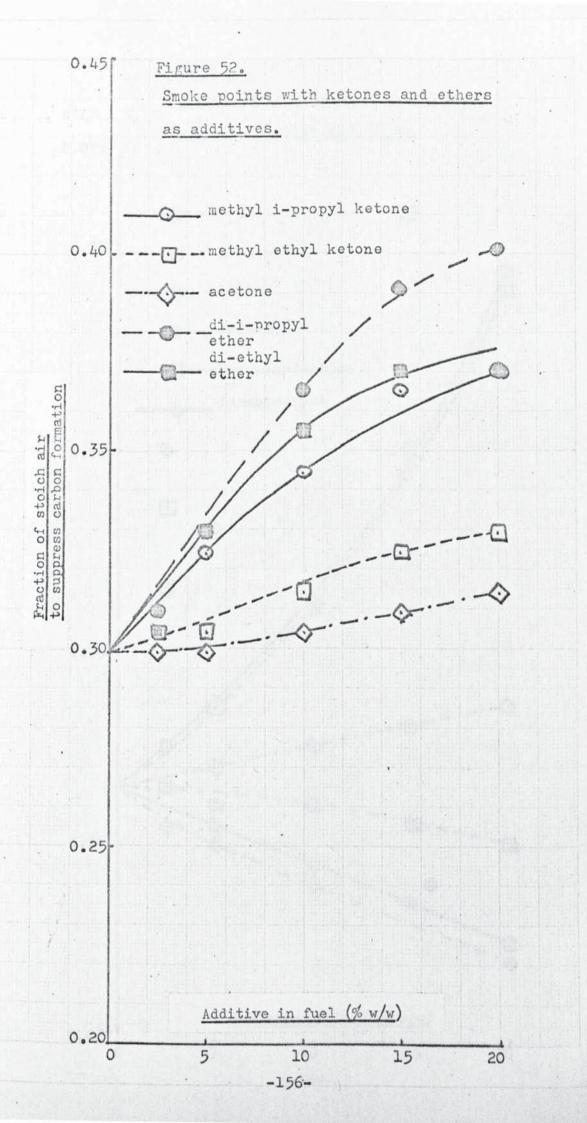
Hydrogen chloride addition resulted in a slight increase in emissivity and trichloro-trifluoroethane addition had the opposite effect, neither being very marked.

Methyl bromide, carbon tetrachloride, chloroform, and, to a lesser extent, hydrogen chloride all had similar maximum temperature-concentration profiles (fig. 49). Low addition levels increased whereas higher levels decreased the maximum flame temperature.

In the cases of the halogenated hydrocarbons the increase in the calorific input to the flame resulted in the higher temperatures at low addition rates. At the higher rates the calorific value of the extra fuel being supplied was outweighed by the consequent increase in the emissivity. The nett result was more heat being radiated from the flame which was, therefore, cooled, quite considerably in the cases of methyl bromide and carbon tetrachloride, which produced flames which were even cooler than those with added aromatics. The trichloro-trifluoroethane had little effect on the temperature of the flame.

The severe cooling at high addition rates resulted in the maximum radiances of the flames with methyl bromide and carbon tetrachloride being well below the maximum radiance of the xylene flames (fig. 50). The carbon tetrachloride flames were again similar to the benzene flames when considering the maximum radiance. Neither hydrogen chloride nor trichlorotrifluoroethane had a marked effect on the radiance.





Figurë 53. Maximum emissivities with alcohols and aldehydes as additives. 0.075 n-propanol 0.07 ethanol methanol -propionaldehyde formaldehyde 0.06 Maximum emissivity • 0.05 0.04 Additive in fuel (% w/w) 0.03 10 20 -157-

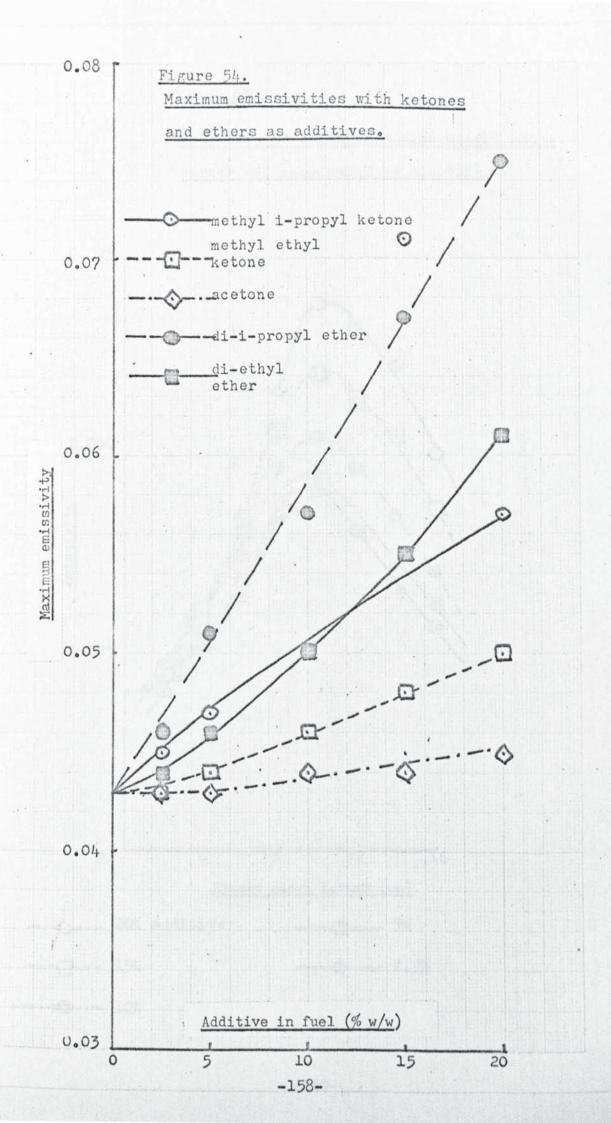


Figure 55.

Variation of emissivity with height above burner for n-propanol as additive.

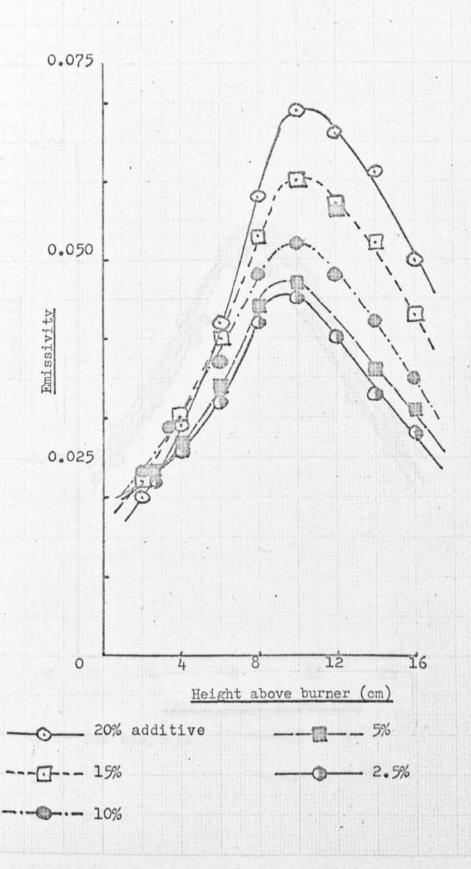


Figure 56.

Variation of emissivity with height above

burner for propionaldehyde as additive.

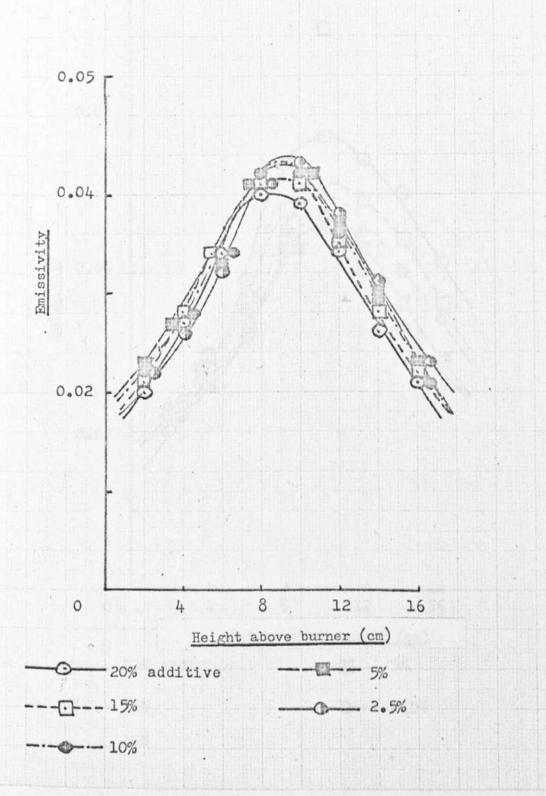
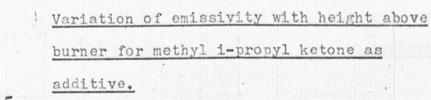


Figure 57.



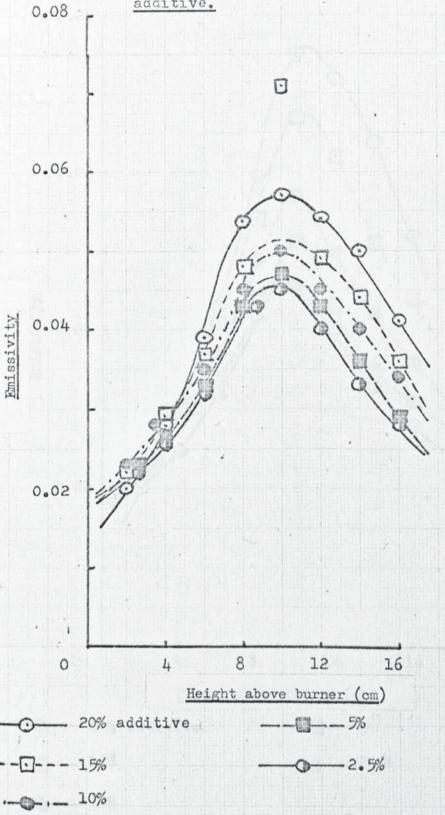


Figure 58.

Variation of emissivity with height above burner for di-i-propyl ether as additive.

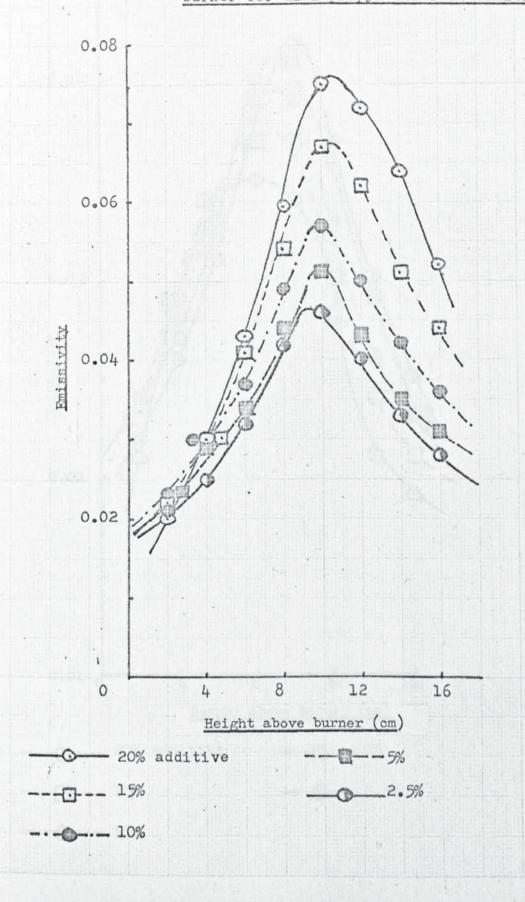


Figure 59. Variation of emissivity with height above 0.045burner for methanol as additive. 0.04 0.03 Emissivity 0.02 0.01 Height above burner (cm)

Figure 60.

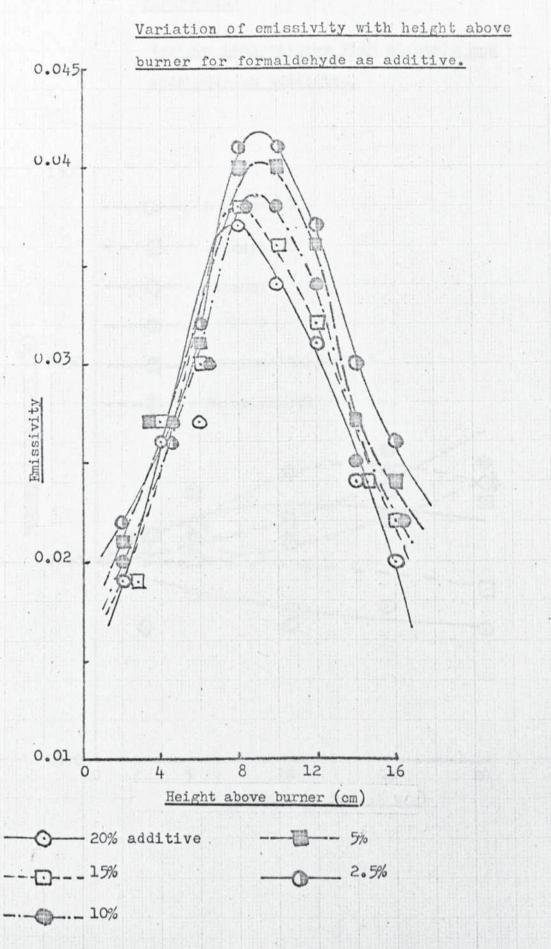


Figure 61.

Maximum temperatures with alcohols and aldehydes as additives.

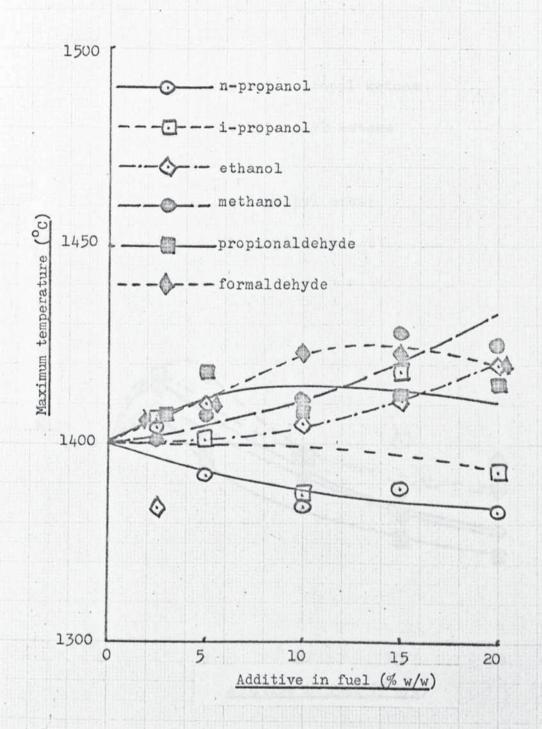


Figure 62.

Maximum temperatures with ketones and ethers as additives.

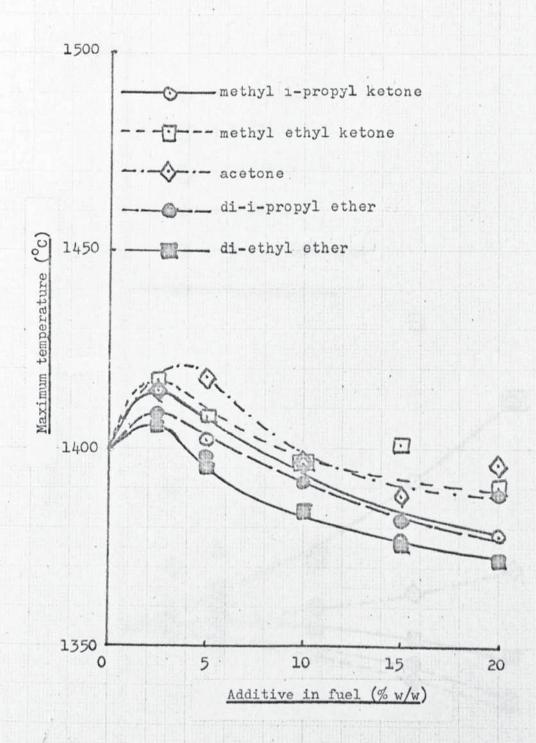
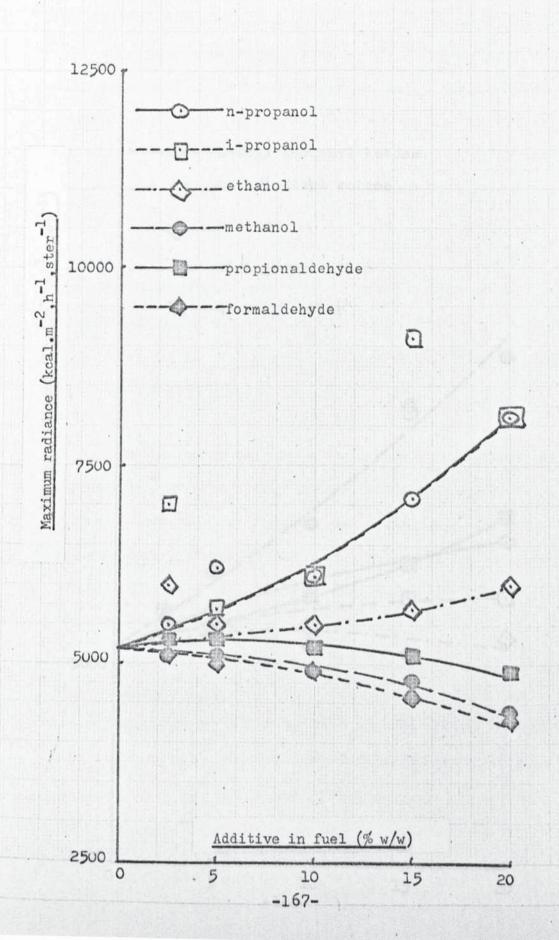
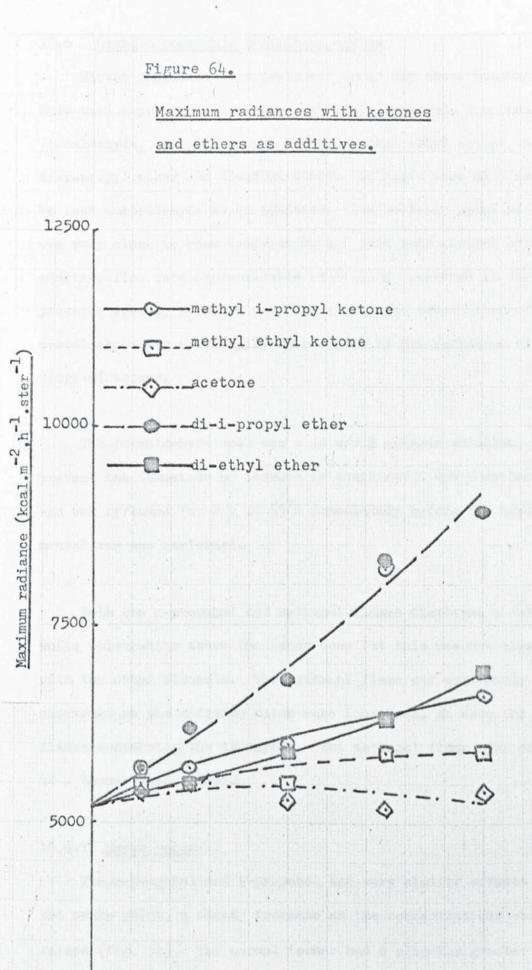


Figure 63.

Maximum radiances with alcohols and aldehydes as additives.





Additive in fuel (% w/w)

-168-

# 10.6 Organic compounds containing oxygen

Eleven additives were evaluated under the above heading. They were n-propanol, i-propanol, ethanol, methanol, propionaldehyde, formaldehyde, methyl i-propyl ketone, methyl ethyl ketone, acetone, di-i-propyl ether and di-ethyl ether. Attempts were also made to test acetaldehyde as an additive. Its boiling point of 20°C was very close to room temperature and this made control of the additive flow rate impracticable as boiling occurred in the pressure vessel, pipework and flowmeter. The smooth flow of the acetaldehyde was continually interrupted by the formation of slugs of vapour.

The formaldehyde used was a 40 w/v % aqueous solution. To prevent the formation of isomers it contained 1 w/w % methanol and was refluxed for 2 h at 65°C immediately before the experimental run was performed.

Both the n-propanol and methanol flames displayed a faint white colouration above the inner cone but this was not observed with the other alcohols. The methanol flame was noticeably shortened as the additive rates were increased, as were the flames containing the aldehydes. The methanol flame also changed to a deeper blue colour.

### 10.6.1 Smoke point

The n-propanol and i-propanol had very similar effects on the smoke point, a steady increase as the concentrations were raised (fig. 51). The normal isomer had a slightly greater influence than the other though neither were as effective as ethane. Ethanol caused a slight increase in smoke point, at 10% concentration and above. With the presence of the hydroxyl group being more significant in the lower molecular weight alcohol, methanol reduced the smoke point quite considerably, the effect being proportionally less as the concentration was increased.

The behaviour of the aldehydes was very similar to that of the methanol (fig. 51). Propionaldehyde reduced the smoke point slightly less and formaldehyde slightly more than methanol. Within each homologous series the higher molecular weights corresponded to the higher smoke points, as with the paraffins. With these compounds containing oxygen, however, the lower members of the series could actually decrease the tendency of the flame to form soot.

The methyl i-propyl ketone raised the smoke point rather less than the propanols though the shape of the smoke point-concentration curve was very similar (fig. 52). The other ketones were must less effective and were similar to ethanol, again the largest increases in smoke point corresponded to the highest molecular weights.

Di-ethyl ether increased the smoke point slightly more than methyl i-propyl ketone and di-i-propyl ether even more so than the propanols (fig. 52). None of these additives were as effective as the gaseous paraffins, however.

### 10.6.2 Emissivity. temperature and radiance

The influences of the alcohols, aldehydes, ketones and ethers on the smoke points and maximum emissivities of the flames were similar (figs. 53, 54). The isomers of propanol were again very much alike and caused a slightly accelerating rise in the emissivity as their concentrations were increased whereas their effect on the smoke point tended to tail off at high concentrations, however. Ethanol wrought a small increase in the emissivity in contrast to methanol which had the opposite effect. Methanol again lay between the two aldehydes, although closer to firmaldehyde than on the smoke point plot. The ketones had a relatively small influence on the maximum emissivity. Di-ethyl-ether was even less effective at low concentrations and only at 15% and above did its emissivities exceed those with methyl i-propyl ketone. Di-i-propyl ketone had the greatest effect, reaching 0.075 at 20% concentration but this was still inferior to the gaseous paraffins. The position of maximum emissivity varied only slightly with the changes in concentration of the propanols, ethanol, propionaldehyde, ketones and ethers (figs. 55, 56, 57, 58). Methanol and formaldehyde, however, both caused the position of maximum emissivity to move nearer to the burner as their concentrations were increased (figs. 59, 60).

The maximum flame temperatures were little affected by the additives (figs. 61, 62). The aldehyde flames tended to be slightly hotter than the others and the additives with the higher molecular weights were cooler but the general variation

from methane was small. The ketones and ethers had an overall cooling effect at 10% concentration and above.

The radiances of the flames containing the propanols and di-i-propyl ether were raised in increasing proportion as the additive flows were increased (figs. 63, 64). The effectiveness of the propanols and di-i-propyl ether as additives lay between methane and ethane. The other alcohols, aldehydes, ketones and ethers had only a small influence on the radiance.

Figure 65.

Smoke points with reactants and products of combustion as additives.

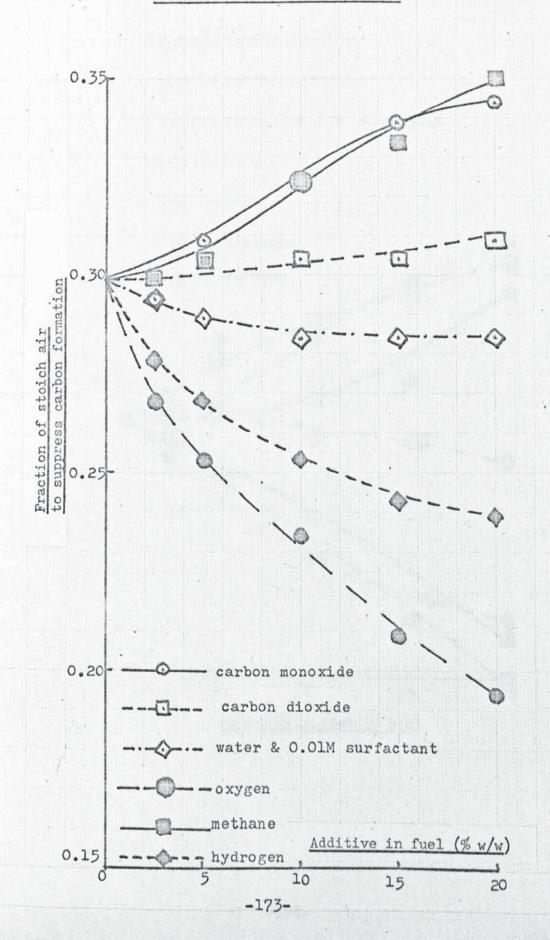


Figure 66.

Maximum emissivities with reactants

and products of combustion as additives.

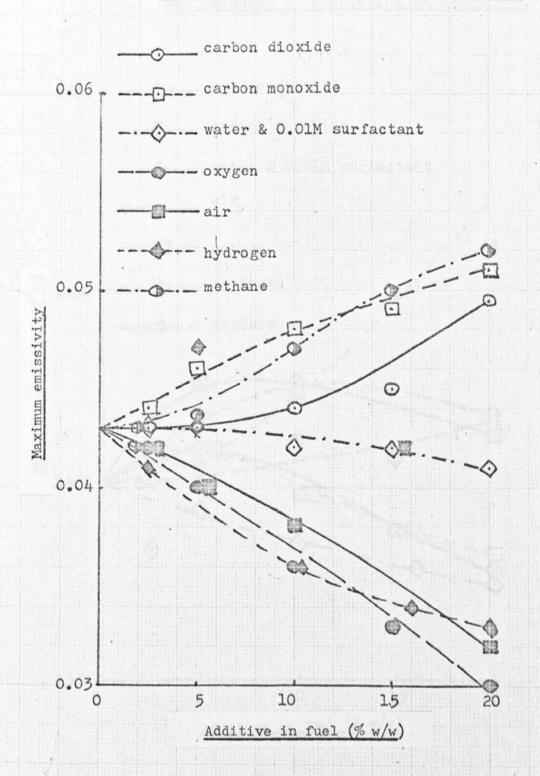


Figure 67.

Maximum temperatures with reactants and products of combustion as additives.

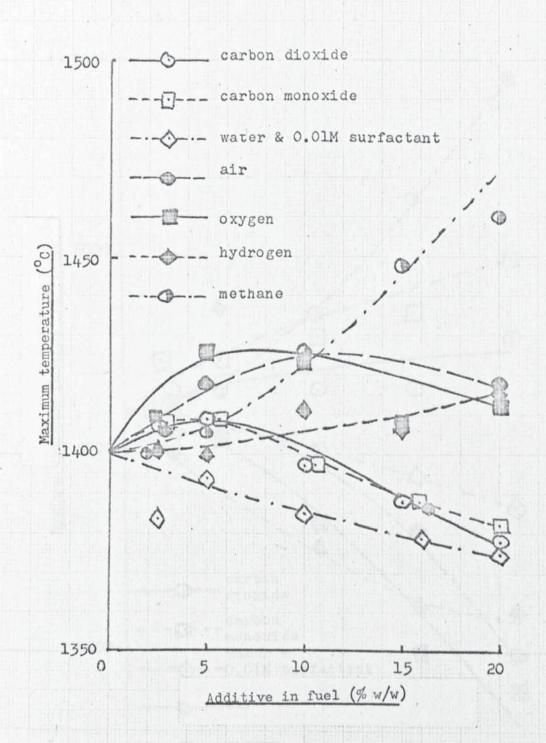


Figure 68.

Maximum radiances with reactants
and products of combustion as additives.

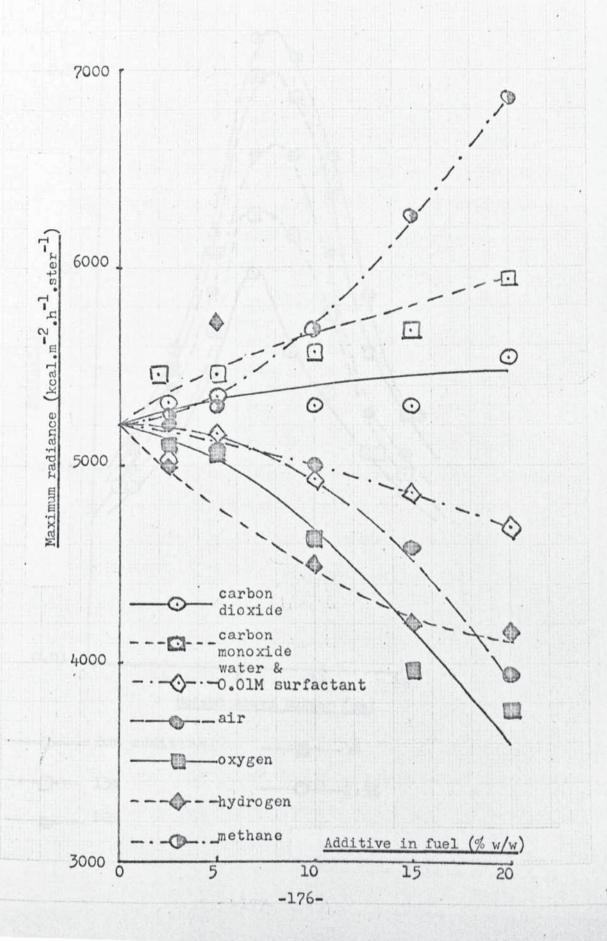
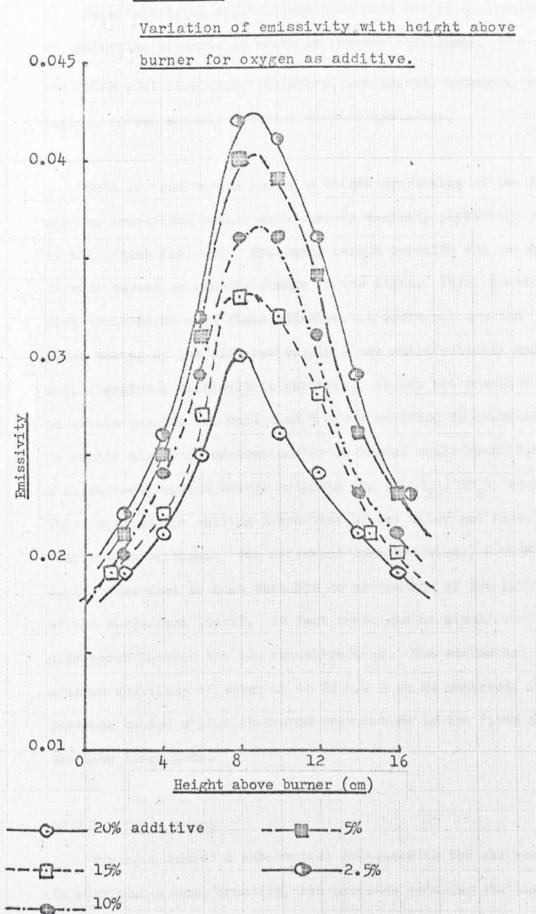


Figure 69.



# 10.7 Flame reactants and products of combustion

Some industrial applications have made use of recirculation of combustion products in order to improve efficiency. The following additives were, therefore, considered, hydrogen, air, oxygen, carbon monoxide, carbon dioxide and water.

Both air and oxygen caused a slight shortening of the flame and the inner cone became more sharply defined, particularly in the latter instance. Hydrogen, carbon monoxide and carbon dioxide caused no visible change in the flame. Water proved very troublesome as a flame additive, it would not wet the glass tubing of the additive supply lines satisfactorily and was, therefore, difficult to vaporise. It was not possible to obtain results for more than 5 w/w % addition of distilled water. To enable higher concentrations to be tested small quantities of a surfactant, sodium lauryl sulphate (Na (C12H25) SO4), were added. This improved the wetting properties of the water and eased the evaporation problems. Two different concentrations, 0.005M and 0.01M, were used to make possible an assessment of the influence of the surfactant itself. In fact there was no significant difference between the two concentrations. The surfactant allowed additions of water up to 20 w/w % to be achieved, its presence caused a pinkish-orange colouration in the flame around the blue inner cone.

### 10.7.1. Smoke point

Hydrogen caused a substantial reduction in the air required to suppress carbon formation, its presence reducing the tendency for dehydrogenation of the methane to radicals to occur (fig. 65). Oxygen was even more effective, a 20% addition reduced the air required by more than 50%, the increased oxidation rate allowing little soot formation. Water and carbon dioxide had only slight effects on the smoke point but carbon monoxide behaved similarly to methane, promoting soot formation. It is probable that the carbon monoxide is to some extent preferentially oxidised to carbon dioxide causing a larger proportion of the methane to form soot than would otherwise be the case. It is unlikely that the carbon monoxide is a part of the soot forming mechanism since its own flame does not form soot (85).

### 10.7.2. Emissivity, temperature and radiance

Air, oxygen and hydrogen all caused reductions in the maximum emissivities of the flames. The reductions were almost linear with concentration (fig. 66). Oxygen was the most effective but not much more so than the others. This was because the contribution to the total emissivity by the soot emissivity was relatively low and so the maximum possible reduction was quite small; at the higher concentrations the total emissivity was approaching that of the gaseous products of combustion alone. In a flame, such as that of a higher paraffin, with a greater tendency to form soot the differences between the air, oxygen and hydrogen would have been more marked.

Water caused a very slight decrease in the maximum emissivity whereas carbon dioxide yielded a small increase. Carbon monoxide was again very similar to methane and the only one of these additives to achieve a significant increase.

None of these additives influenced the maximum flame temperature to a great extent (fig. 67). Water and the oxides of carbon reduced the maximum temperature a little at high concentrations whereas oxygen, air and hydrogen had the opposite effect. 5% oxygen caused the largest increase but this was not maintained at high addition rates.

The maximum radiances of the flames were not improved by any of these additives as much as by methane. The lower emissivities caused by the air, oxygen and hydrogen flames more than outweighed the slight increases in temperatures and resulted in reduced radiances (fig. 68).

These results indicate that the recirculation of combustion products is unlikely to yield large increases in the flame emissivity by virtue of their chemical composition, any improvement being brought about by carbon monoxide which would only be present in flue gases in significant quantities in a rather inefficient furnace. It is quite possible, however, that the high temperature of recirculated combustion products could have a greater influence by cracking of the methane.

Smoke points with miscellaneous additives.

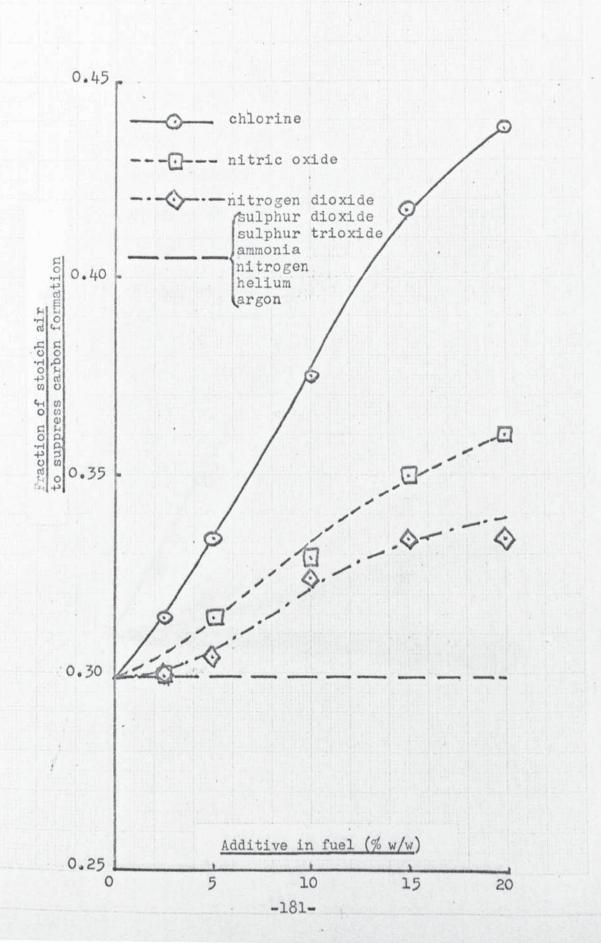
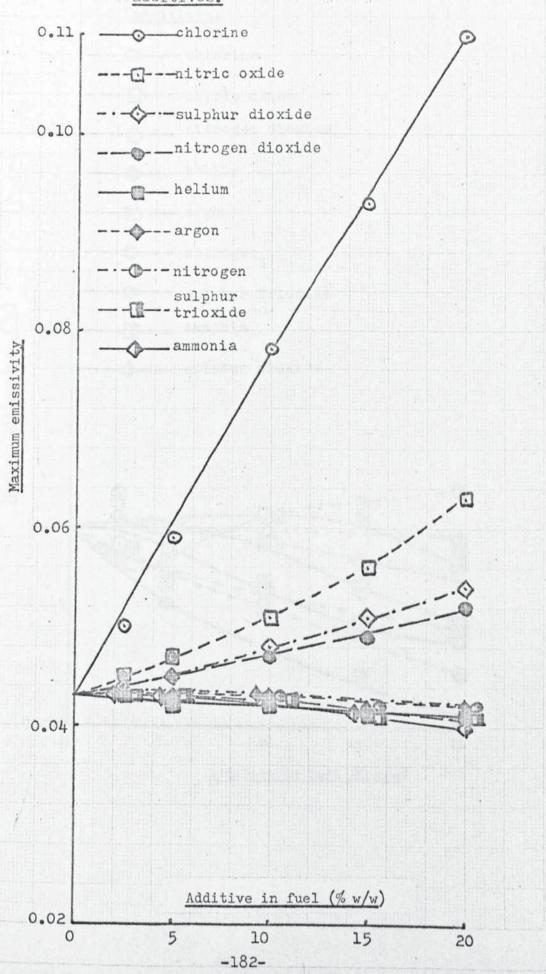


Figure 71.

Maximum emissivities with miscellaneous additives.



# Figure 72.

# Maximum temperatures with miscellaneous additives.

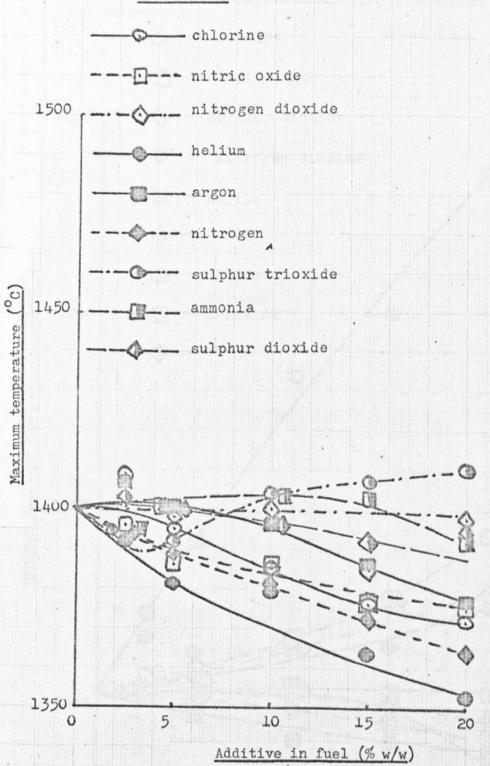
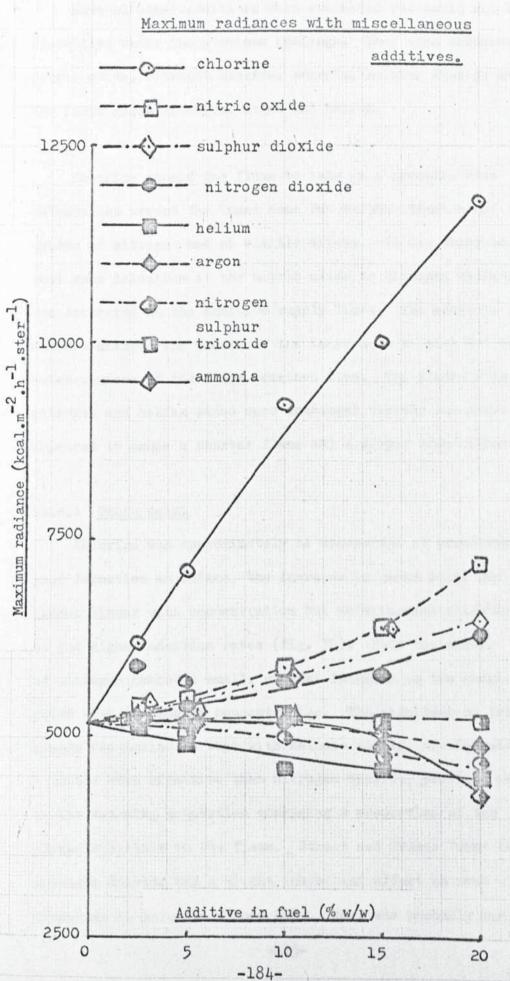


Figure 73.



### 10.8 Miscellaneous additives

Several other additives were evaluated but could not be classified under the previous headings. They were chlorine, nitric oxide, nitrogen dioxide, ammonia, sulphur dioxide and the inert gases nitrogen, argon and helium.

Chlorine caused the flame to take on a greenish blue colouration around the inner cone but sulphur dioxide and the oxides of nitrogen had no visible effect. It was observed that some oxidation of the nitric oxide to nitrogen dioxide was occurring in the additive supply lines. The addition of ammonia changed the colour of the inner cone to pink but the outer regions of the flame remained blue. The flames with nitrogen and helium added were unchanged visibly but argon appeared to cause a shorter flame and a deeper blue colour.

#### 10.8.1 Smoke point

Chlorine was approximately as successful at promoting soot formation as butane, the increase in smoke point was almost linear with concentration but deteriorated slightly at the higher addition rates (fig. 70). Both the oxides of nitrogen caused a small, gradual increase in the smoke point with increasing concentration. The magnitude of the change was similar to that with methane itself, nitric oxide was a little more effective than nitrogen dioxide, possibly due to its reducing properties consuming a proportion of the oxygen available in the flame. Street and Thomas found that nitrogen dioxide had a slight inhibiting effect on soot formation in kerosene flames (85). This was probably due

to the dissociation of the nitrogen dioxide to nitric oxide and oxygen,

the oxygen could then have its normal inhibiting effect.

The lower temperatures of the small methane flame would have reduced the degree of dissociation.

The inert gases and sulphur dioxide had no significant effect on the smoke point. Street and Thomas found that nitrogen had a promoting influence on soot formation but they used much larger quantities, effectly reducing the oxygen content of the 'air' to as low as 13 v/v %. A nitrogen addition of 20 w/w % as defined earlier corresponds approximately to a reduction of the oxygen in the premixed air from 21 v/v % to 20.5 v/v %. The sulphur dioxide result agreed with that of Street and Thomas in contradiction to the work of Gaydon and Whittingham (74) who found that there was a suppression of the smoke point. The claim of the former that the quantity of soot formed was reduced was not confirmed, as will be seen the results for emissivity suggest that this was not the case.

# 10.8.2 Emissivity, temperature and radiance

Chlorine had a considerable effect on the maximum emissivity of the flame causing a linear increase with concentration reaching an emissivity of 0.11 at 20 w/w %

addition (fig. 71). As with the smoke point the behaviour was very similar to butane although the lower addition rates had slightly more effect than with the paraffin. The addition of the chlorine moved the point of maximum emissivity away from the burner but as with the other soot promoting additives the high values were achieved over short lengths of flame only.

Nitric oxide, nitrogen dioxide and sulphur dioxide all caused small increases in the maximum emissivity whereas the inerts and ammonia brought about very slight decreases. There was no significant variation among the inert gases.

Changes in the maximum flame temperature arising from
the use of these additives were generally small and they all
had a cooling effect (fig. 72). The degree of cooling by
the inerts was inversely proportional to their molecular
weights, helium having a greater influence than nitrogen
which was in turn more effective than argon. This may have
been due, in part, to the higher velocities of the combustion
mixtures with the lighter gases creating turbulence and a more
homogeneous flame without regions of extreme temperature. The
Reynold's Number is not changed by the use of the different
inerts, of course, and further work will be necessary to fully
explain this phenomonon. The high emissivity of the chlorine
flame in the absence of any additional input to compensate
resulted in lower temperatures than for butane and the other
paraffins.

The lower temperatures meant that the chlorine flames actually had lower maximum radiances than the ethane flames even though their emissivities were similar to those of flames with butane (fig. 73). This would not necessarily occur to the same extent with larger industrial scale flames. Sulphur dioxide and the oxides of nitrogen all achieved small increases in the maximum radiance. Ammonia lowered the radiance slightly, this was most marked at 15 w/w % and 20 w/w % concentrations. The inerts had little significant influence.

### 10.9 Sulphur trioxide

Following the work of Whittingham, Gaydon and Dooley

(74, 75, 76) on sulphur trioxide with methane, ethane and

coal gas and the work of Street and Thomas (85) on sulphur trioxide

with bensene and kerosene flames it was anticipated that this

additive would greatly enhance soot formation in the flame.

The sulphur trioxide was added to the flame by passing a suitable fraction of the air supply through a flask of oleum at the appropriate concentration and temperature for each addition rate to be achieved. The method was similar to that used by Gaydon and Whittingham.

It was found that the addition of sulphur trioxide in quantities up to 20 w/w % of the methane had no significant effect on the visual appearance of the flame nor on the measured quantities. This result was unexpected and so even larger addition rates, up to 40 w/w %, were tested but again with no increase in soot formation.

At this stage it was decided to determine the effect of sulphur trioxide on a flame which was already on the point of forming soot. A richer flame than normal was established (air flow 0.088 l.s<sup>-1</sup>, methane flow 0.0238 l.s<sup>-1</sup>) and sulphur trioxide was added, again with no improvement in the radiating properties. The sulphur trioxide was added to the methane instead of the air but this was also unsuccessful. The qualitative effects of these and other experiments involving sulphur trioxide are summarised in Appendix 6.

In order to ensure that the sulphur trioxide was being added satisfactorily and also to check some of the earlier work a town gas supply was run to the test rig, the gas was temporarily injected directly into the burner mixing chamber. A non-luminous town gas flame was set up and sulphur trioxide was added to the air supply. Soot formation was observed immediately. The appearance of the flame was quite different to that of the other luminous flames observed in that the yellow colouration due to the soot was present right from the base of the flame at the rim of the burner. At this time no measurements were made on the flame. It was thought that the sulphur trioxide was causing one of the hydrocarbons, other than methane, present in the town gas to form soot, ethane and propane were the only hydrocarbons which could be detected. It was decided, therefore, to contaminate the normal methane supply with these gases in turn to confirm or deny this theory. The sulphur trioxide was found to have no effect on these mixtures of methane and ethane or methane and propane. The problem of which component of the town gas was taking part in the soot formation remained. A simulated town gas was made up from pure components in an attempt to establish whether one of the major components was responsible or not. The composition was:

co <sub>2</sub>	12.4	v/v %
CO	1.5	
H <sub>2</sub>	53.6	
N <sub>2</sub>	0.5	
CH <sub>4</sub>	31.2	
c <sub>2</sub> H <sub>6</sub>	0.7	
<sup>C</sup> 3 <sup>H</sup> 8	0.1	
	100.0	v/v %
	-190-	

This gas was fed to the rig at the normal methane inlet point and therefore passed through the carbon dioxide and moisture absorber. The addition of sulphur trioxide caused no change in the visual appearance of the flame, indicating that the soof formed in the genuine town gas flame was caused by a trace component.

It was considered necessary to make some measurements of this descrepancy so the town gas was reconnected but this time at the methane inlet, prior to the absorber, and not directly at the burner. On this occasion not even the genuine town gas was affected by the addition of the sulphur trioxide.

Rechecking, the town gas was injected at the burner and the yellow colouration was observed again throughout the flame.

Admitting the simulated town gas at this point yielded no increased soot formation. It was clear that the trace component of the town gas which caused soot formation with sulphur trioxide was absorbed by either silica gel or carbosorb. The town gas flow was 0.0326 l.s<sup>-1</sup>, the air flow was 0.0351 l.s<sup>-1</sup>.

The carbosorb was removed from the absorber and town gas was passed through again, there was noyellow colouration when sulphur trioxide was added. The silica gel was replaced by carbosorb and the yellow colouration returned. The important trace component was, therefore, absorbed by silica gel and so could have been water. Town gas was passed through silica gel and then bubbled through water while sulphur trioxide was added to the air, there was no soot formation indicating that water was not the trace component.

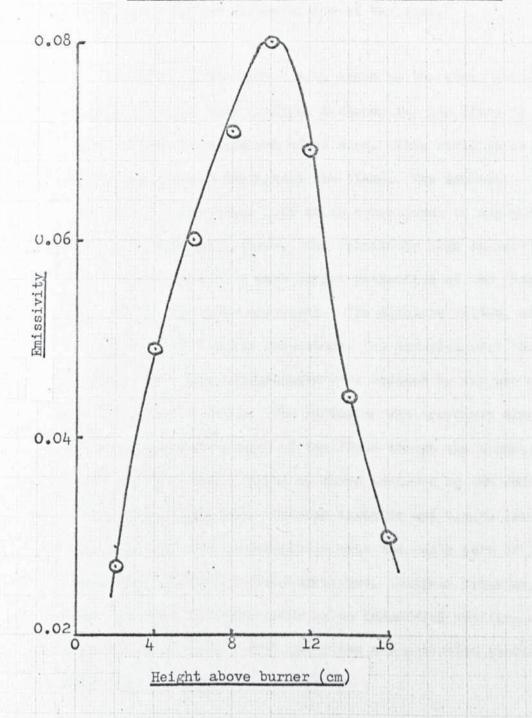
Three other possibilities were considered at this stage.

The town gas supply to the laboratory originated at a gas works using I.C.I. steam-naphtha reformers and Gas Council recycle hydrogenators. The rich gas produced by these hydrogenators is passed through a benzene wash to remove aromatics such as naphthalene, the town gas could, therefore, have contained a trace of benzene and it was known that sulphur trioxide caused copious quantities of soot in benzene flames (85). The final stage of the gas-making process is a drying plant utilising triethylene glycol as an absorbant. The town gas could have been containinated by small quantities of triethylene glycol.

Before the town gas is distributed a small proportion of odorant is added since it has no inherent smell. Usually about 1 lb/10<sup>6</sup>sft<sup>3</sup> of tetra-hydrothiophene (t.h.t.) is used and some of this was present in the town gas used in the tests.

The town gas was again passed through the silica gel to remove the important component and then through benzene, triethylene glycol and t.h.t. in three different experiments. All three gave slight yellow colourations at the flame tip, even without sulphur trioxide. When sulphur trioxide was added to the air supply there was no significant change in the flames with benzene or triethylene glycol but with t.h.t. there was a strong yellow colouration throughout the flame just as with the untreated town gas.

This evidence suggests that the soot formation caused by the addition of sulphur trioxide to coal gas, previously observed by others, was due to the presence of a compound or compounds Variation of emissivity with height above burner for sulphur trioxide (20% w/w) and tetrahydrothiophene (trace) as additives.



similar to t.h.t.. At the time of the previous work,

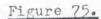
1945 - 1947, there was no oil reformed gas and no added

odorant but the coal gas would have included sulphur containing

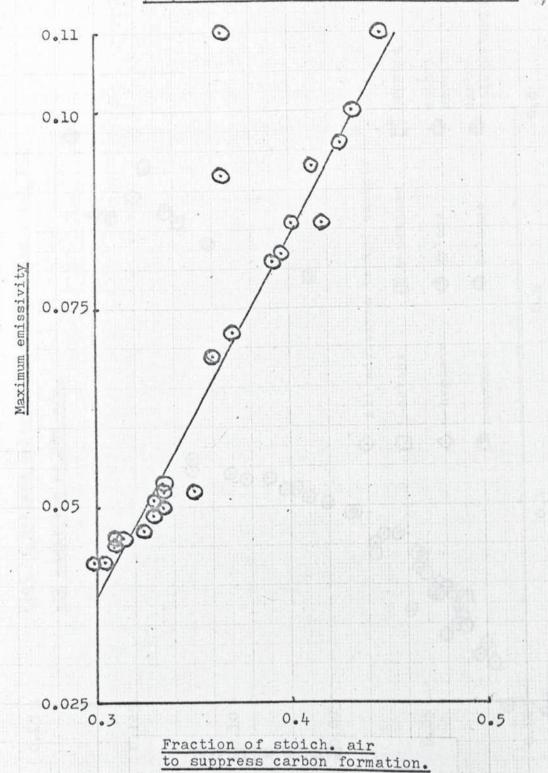
compounds similar to t.h.t., e.g. thiophene, dihydrothiophene,

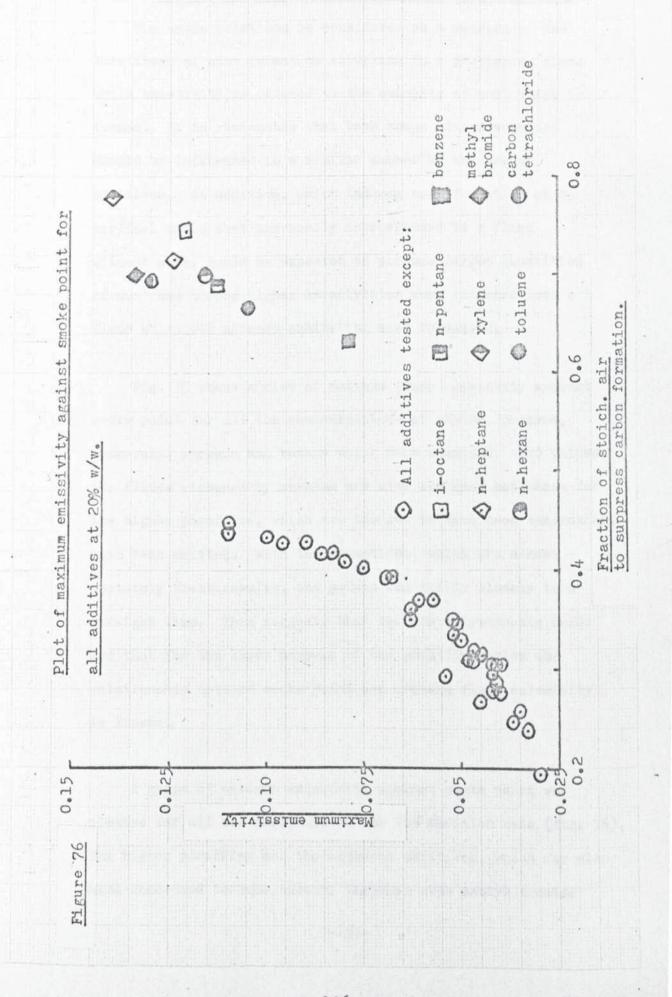
produced during the carbonisation of the coal.

Tests were made with t.h.t. added to the usual methane flame, there was only a slight dendency for the flame to soot until sulphur trioxide was added also. This resulted in a yellow colouration throughout the flame. The maximum emissivity recorded was 0.08 which corresponds to approximately 0.8 in an industrial flame. The relatively high emissivities were maintained over a much larger proportion of the flame than with any of the other additives. The absolute values, however, were not as high as for iso-octane, for example, over this length. The flame temperatures were reduced by the use of the two additives together. The radiation was increased almost uniformly over the length of the flame though the highest increases were not as great as those achieved by the addition of paraffins (fig. 74). Sulphur trioxide and t.h.t. achieved a greater increase in emissivity over the early part of the flame than any of the other additives. Sulphur trioxide. however, would be undesirable as an industrial additive due to handling difficulties with the oleum and pollution problems with the flue gases.



Plot of maximum emissivity against smoke point for methane, ethane, propane, commercial propane and butane at all concentrations.





# 10.10 The relationship between smoke point and emissivity

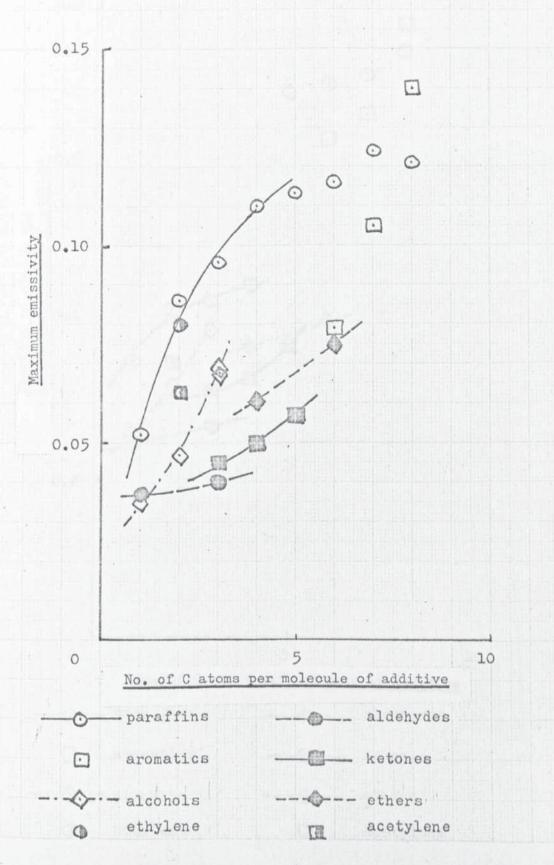
The smoke point can be considered as a measure of the likelihood of soot formation occurring in a particular flame while emissivity is related to the quantity of soot which is formed. It is reasonable that both these characteristics should be influenced in a similar manner by the use of additives. An additive, which induces soot formation at an air/fuel ratio that previously corresponded to a flame without soot, could be expected to produce larger quantities of soot and thence higher emissivities when injected into a flame which was already exhibiting soot formation.

Fig. 75 shows a plot of maximum flame emissivity against smoke point for all the concentrations of ethane, propane, commercial propane and butane which were examined. The values for flames richened by methane are also included but those for the higher paraffins, which are thought to have been condensing, have been omitted. With two exceptions, which are almost certainly freak results, the points lie fairly closely to a straight line. This suggests that the above hypothesis holds and that for the lower members of the paraffin series the relationship between smoke point and maximum flame emissivity is linear.

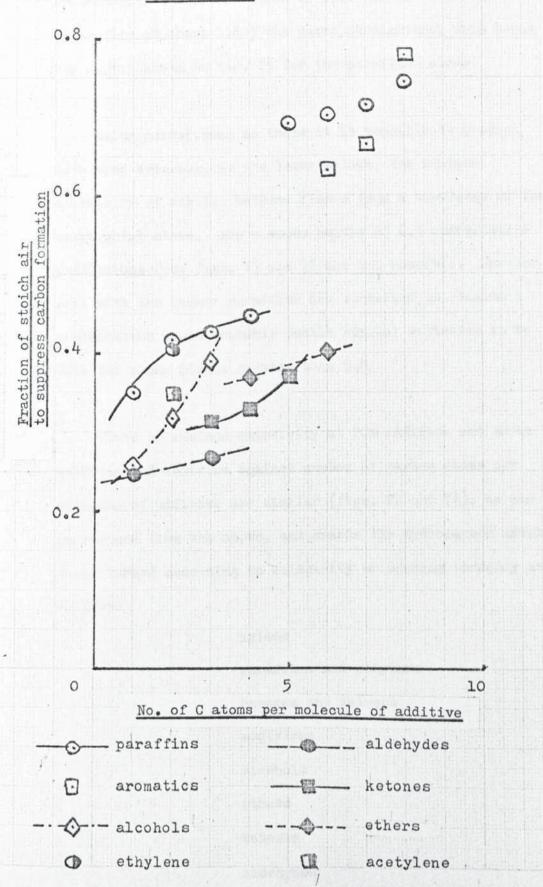
A graph of maximum emissivity against smoke point was plotted for all the additives at the 20% addition rate (fig. 76). The higher paraffins and the aromatic additives, which may also have condensed to some extent, together with methyl bromide

Figure 77.

Plot of maximum emissivity at 20% w/w addition against number of C atoms per molecule of additive.



Plot of smoke point at 20% w/w addition against number of C atoms per molecule of additive.



and carbon tetrachloride do not show the same behaviour as the rest of the additives which lie approximately on a smooth curve. The corresponding points for the other concentrations also lie close to this curve. Above an emissivity of about 0.055 the curve straightens, this being the region shown in fig. 75 for the paraffins alone.

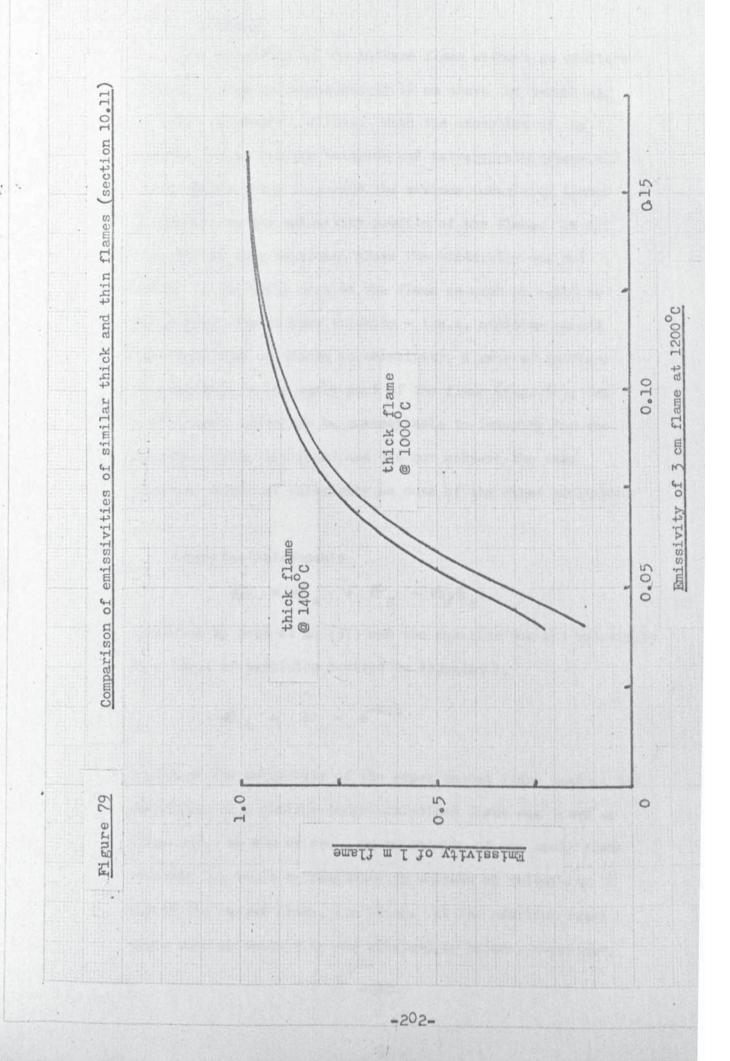
Using curves such as these it is possible to predict with some accuracy, at the lower values, the maximum emissivity of similar methane flames from a knowledge of the smoke point alone. Above smoke points of 0.5 quantitative predictions from figs. 75 and 76 are not possible. Further work with the higher paraffins and aromatics to eliminate condensation would probably enable similar estimates to be made for smoke points greater than 0.5.

Plots of maximum emissivity at 20% addition and smoke point at 20% addition against number of carbon atoms per molecule of additive are similar (figs. 77 and 78), as can be deduced from the above, and enable the hydrocarbon additives to be ranked according to emissivity or sooting tendency as follows:

xylene
paraffins and ethylene
benzene and toluene
acetylene
alcohols
ethers
ketones
aldehydes

Of the hydrocarbons tested only methanol, formaldehyde and propionaldehyde caused a decrease in maximum emissivity and a lowering of the smoke point.

For estimating the influence of additives on the emissivity of a flame, the feasibility of measuring the smoke point as a first approximation of the relative effects, instead of measuring the emissivity itself, will be of considerable benefit.



#### 10.11 Scale-up

The emissivity of the methane flame without an additive rose to a peak at approximately 10 cm above the burner and declined at greater heights. With the exception of the combination of sulphur trioxide and tetrahydrothiophene all the additives which increased the maximum emissivity tended to exaggerate the emissivity profile of the flame. In all but the one case mentioned above the emissivity was not raised in the early part of the flame as much as would be desirable. The sulphur trioxide - t.h.t. addition caused the right kind of change in emissivity, a general increase particularly in the early part of the flame (fig. 74), but would almost certainly be unacceptable to industry for the reasons already mentioned and did not achieve the same absolute values of emissivity as some of the other additives.

Using the relationship

developed by Sato et al (37) and the equation for the emissivity of a cloud of particles derived in Appendix 1.

a plot of the emissivity of the experimental flame against the emissivity of a similar industrial-sized flame was drawn up (fig. 79). As can be seen, an emissivity of the small flame of about 0.1 would be necessary to achieve an emissivity of 0.9 in the larger flame, 1 m thick. At the addition rates used, such an emissivity was attained by butane, n-pentane,

n-hexane, n-heptane, i-octane, toluene, xylene, methyl bromide, carbon tetrachloride and chlorine. The flame was considered to be a maximum of 16 cm long though this varied with the additive being used. Xylene and methyl bromide gave the greatest length of flame above an emissivity of 0.1, approximately 7 cm in both cases, slightly less than half the total and falling short of the 75 - 100% of the flame length which was considered desirable.

The higher paraffins and the aromatic hydrocarbons caused lower maximum flame temperatures than the lower paraffins. This was partly due to the relatively high emissivities leading to increased radiation from the flame and consequent cooling. In addition the efficiency of combustion of the flame was reduced by the formation of the soot and some of the heating potential of the flame was not realised, again resulting in lower temperatures. The latter effect was exaggerated by the condensation of some of the additives, the thermal cracking of the droplets probably resulted in larger soot particles than those formed according to a synthesis mechanism and the ultimate combustion of these particles would have been less likely with a resultant loss in the heat released in the flame. This cooling would occur in similar industrial-sized flames but it can be seen that for all these additives the simultaneous increase in emissivity was sufficient to yield a net increase in radiance compared with the pure methane flame.

#### 11. Conclusions

- The following additives were found to increase the maximum emissivity of the methane flame: ethane; propane; butane; n-pentane; n-hexane; n-heptane; i-octane; benzene; toluene; xylene; acetylene; ethylene; cyclohexane; chloroform; carbon tetrachloride; methyl bromide; ethanol; n-propanol; i-propanol; methyl ethyl ketone; methyl i-propyl ketone; di-ethyl ether; di-i-propyl ether; carbon monoxide; carbon dioxide; nitrogen dioxide; sulphur dioxide; nitric oxide; chlorine and a combination of sulphur trioxide with tetrahydrothiophene.
- 2. The following additives caused an increase in the air required to prevent soot formation in the flame: ethane; propane; butane; n-pentane; n-hexane; n-heptane; i-octane; benzene; toluene; xylene; acetylene; ethylene; cyclohexane; chloroform; carbon tetrachloride; methyl bromide; ethanol; n-propanol; i-propanol; methyl ethyl ketone; methyl i-propyl ketone; di-ethyl ether; di-i-propyl ether; carbon monoxide; carbon dioxide; nitrogen dioxide; nitric oxide and chlorine.
- 3. The following additives decreased the maximum emissivity of the methane flame: methanol; formaldehyde; propionaldehyde; oxygen and hydrogen.

- 4. The following additives caused a reduction in the air necessary to prevent soot formation: methanol; formaldehyde; propionaldehyde; water; oxygen and hydrogen.
- 5. The following additives had no significant influence on the maximum emissivity of the methane flame: hydrogen chloride; trichloro-trifluoroethane; acetone; water; helium; nitrogen; argon; ammonia and sulphur trioxide.
- 6. The following additives caused no significant change in the air necessary to prevent soot formation: hydrogen chloride; trichloro-trifluoroethane; acetone; helium; nitrogen; argon; ammonia; sulphur dioxide and sulphur trioxide.
- 7. The magnitudes of changes in emissivity and air required to suppress soot formation are dependent not only upon the quantity of additive used but also on the molecular structure of the additive. The organic compounds tested may be ranked in order of decreasing effectiveness as follows:

xylene

paraffins and ethylene

benzene and toluene

acetylene

alcohols

ethers

ketones

aldehydes

- 8. The highest members of the homologous series produced
  the largest increases in emissivity and critical air
  concentration when used as additives, though not necessarily
  the largest increases in radiance.
- 9. In the paraffin series the difference in effectiveness between adjacent homologues is more marked at the lower end than at the high end.
- 10. The addition of ethane or higher paraffins has a much greater influence than supplying extra methane to the flame.
- 11. The increases in emissivity and critical air concentration caused by the addition of alkyl halides are greater for higher degrees of halation of the additive.
- 12. Brominated hydrocarbons have a greater influence than chlorinated hydrocarbons.
- 13. The greatly increased soot formation in coal gas and town gas flames after the addition of sulphur trioxide is due to reaction between the latter and traces of organic compounds containing sulphur which are present in these fuels.
- 14. Those additives which cause the greatest increases in emissivity also reduce the temperatures of the flames but there are, nevertheless, net gains in radiance.

- 15. None of the additives tested produced a flame which would be entirely suitable for industrial use. Satisfactory peak emissivities of 0.1 were obtained by the use of butane, n-pentane, n-hexane, n-heptane, i-octane, toluene, xylene, methyl bromide, carbon tetrachloride and chlorine but such high levels were not maintained over a large fraction of the flame. The mixture of sulphur trioxide and tetrahydrothiophene was the only additive to give relatively large increases in the emissivity in the early part of the flame, the absolute values were not sufficiently high, however.
- 16. The use of additives affects the maximum emissivity and the air concentration necessary for suppression of soot formation in similar ways. The relationship between the two effects is linear for the lower paraffins.
- 17. This makes it possible for experimentally determined values of the critical air concentration to be used for predicting the relative effects of additives on the emissivity.

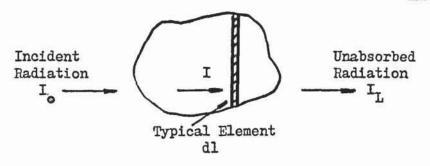
#### 12. Recommendations for future work

Since the commencement of this work, considerable effort has been applied to the design and development of the two-stage precracking burners referred to in section 2.3.2.. These have proved successful and are being adopted for a wide range of applications so that the need for additives to raise the emissivities of flames has been reduced. The following work would be of interest however:

- Evaluation of the effects of higher aromatic compounds such as derivatives of naphthalene, e.g. tetralin, decalin. 1-methyl naphthalene.
- 2. Checks on the accuracy of the predictions of the scale-up of emissivity, preferably using a small furnace with a flame thickness of at least 0.5 m.
- 3. Experiments on the effects of the paraffins on a larger flame to establish if the gains in radiance are less with the higher members of the series, as occurred in the small flame.
- 4. Further experimentation with the liquid paraffins, taking further precautions to eliminate condensation in the burners, to find if the limits of the smoke point - emissivity relationship can be extended beyond butane.
- 5. Further experimentation on the effects of the addition of inert gases on the aerodynamic properties of flames with particular emphasis on their stability.



The emissivity of a cloud of particles



$$\frac{dI}{I} = -A_{p}. \frac{\text{Area of particles @ 90}^{\circ} \text{ to radiation}}{\text{Area of element @ 90}^{\circ} \text{ to radiation}}$$

$$= -A_{p}. \frac{T \cdot \frac{d_{p}}{2}}{4} \cdot (N_{p}/\text{volume of element})$$
Area of element

but volume element = thickness of element, dl

$$\frac{dI}{I} = \frac{-A_p \cdot \pi \cdot d_p^2}{4} \cdot N_p \cdot dI$$

$$\frac{dI}{I} = \frac{-A_p \cdot \pi \cdot d_p^2}{4} \cdot \frac{d1 \cdot 6 \cdot c}{(\pi \cdot d_p)^3}$$

For a finite thickness L,

$$\int_{0}^{L} \frac{dI}{I} = -\int_{0}^{L} \frac{3}{2} \cdot \frac{A_{p} \cdot c}{\rho \cdot d_{p}} dI$$

$$\ln I - \ln I_{o} = -\left[\frac{3}{2} \cdot \frac{A_{p} \cdot c}{\rho \cdot d_{p}}\right]_{0}^{L}$$

$$\frac{\ln \underline{I}}{I_o} = -\frac{3}{2} \cdot {}^{A_p} \frac{\underline{c}}{\rho} \cdot \frac{\underline{L}}{\underline{d}_p}$$

For particles much smaller than the wavelengths being considered, it has been shown that scattering of the radiation may be disregarded (39, 40, 41, 42, 49). The average diameter of soot particles has been shown to be about 0.04 (38) compared to the significant region of the infra red from approximately 1.0 to 15 % when considering heat transfer. Ignoring the effects of scattering will not in this case incur serious error. Furthermore, it has been shown that the possibility of reflection by soot particles may also be disregarded (43, 44, 45, 46, 47). Hence for soot particles,

$$A_p = E_p = 1$$

and for the cloud as a whole

$$\epsilon_s = 1 - \frac{I}{I_o}$$

$$\epsilon_s = 1 - \exp(-\frac{3}{2}, \frac{c}{p}, \frac{L}{d_p})$$

#### Appendix 2

The following computer program was used for curve fitting.

```
DIMENSION X(200), Y(200), R(200), W(200),
               A(21), P(510)
     INTEGER WTS, STORE (21)
     READ (1,100) M,NMAX,WTS
 100 FORMAT (310)
     TEST FOR NUMBER OF DATA POINTS>200
C
     IF (M-200) 4, 4, 5
     WRITE (2,200)
 200 FORMAT (/49 H ARRAYSIZE OF X, V, R AND W
    10NLY DIMENSIONED TO 200, /60H THE FIRST
    2200 POINTS HAVE THEREFORE BEEN TAKEN AND
    3ANALYSED, /)
     M=200
C
   TEST FOR DEGREE 20
C
C
     IF (NMAX -20) 1, 1, 2
 4
C
     INCREASE NMAX BY 1 TO FORM NEW NMAX
C
 1
     NMAX = NMAX + 1
     GO TO 3
C
C
     NMAX>OR EQUAL TO 20, NEW NMAX SET TO 21
C
     OR M WHICHEVER IS THE SMALLER
C
     NMAX = 21
     NMAX = MINO (NMAX,M)
     MM = 5* NMAX + 2*M + 3
 3
C
C
     TEST FOR FORMAT 1 OR 2
C
     IF (WTS -2) 10, 11, 10
    DO 6 I = 1,M
 11
C
C
     READ IN DATA POINTS + FREQUENCY
     READ (1,101) X (I), Y(I), W(I)
 101 FORMAT (3FO.0)
     CONTINUE
     GO TO 140
 10 DO 12 I=1,M
     READ IN DATA POINTS
C
 12 READ (1,101)X(I),Y(I)
 140 P(3) = 0.0
     P(4) = 1.0

P(5) = WTS-1
     P(1) = -1.0
```

```
SUMX2 = 0
      SUMY = 0
      SUMX = 0
     DO 17 I=1,21
 17 STORE (I) = I-1
     DO 15 I=1,M
C
C
      SUMMATION OF X AND Y POINTS
C
     SUMX = SUMX + X(I)
     SUMY = SUMY + Y(I)
     CONTINUE
 15
C
     CALCULATION OF MEAN X AND MEAN Y
C
C
     SUMX = SUMX/M
     SUMY = SUMY/M
     WRITE (2,206) SUMX, SUMY, M
 206 FORMAT (1H1,////,8H MEANX = ,E21.12,
    14X,7H MEANY = ,E21.12, //, I4, 23H
    2PAIRS OF X AND Y INPUT)
     IDEG = NMAX -1
     WRITE (2,201) IDEG
 201 FORMAT (41HO COEFFS FOUND FOR A
    1POLYNOMIAL OF DEGREE ,13,//)
C
     CHECK FOR OVERFLOW
C
     CALL OVERFL(I)
     IF (I-1) 155, 155, 156
 155 WRITE (2,225)
 255 FORMAT (//,19H** OVERFLOW SET ***,//)
 156 DO 7 I=1,NMAX
C
     SCIENTIFIC SUBROUTINE CALLED IN A LOOP
C
     FROM 1 to NEW NMAX
     CALL F40FORPL (M, NMAX, MM, X, Y,
    lR,W,A,P)
C
C
     DEGREE O INFORMATION NOT CALCULATED
C
     IF (I-1) 7,7,16
 16 \quad SUMSQ = 0.0
   DO 13 N=1,M
     YIJ = A(1)
     DO 14 II = 2,I
     IF (X(N)-0.0)8,14,8
     YIJ = TIJ + A(II) *X(N)**(II-1)
 8
C
     CALCULATION OF Y FROM CURRENT EQUATION
C
     USING EACH X(INPUT)
 14 CONTINUE
C
C
     SUMMATION OF Y(INPUT) - Y(CALCUIA TED) **2
     SUMSQ = SUMSQ+(Y(N)-YIJ)*(Y(N)-YIJ)
 13 CONTINUE
     J=I-1
```

```
WRITE (2,207)J,SUMSQ

207 FORMAT (//,8H DEGREE,13,/,11H ----
1----,/,27H SUM OF SQUARED

2DEVIATIONS = ,E21,12,/)

WRITE (2,203)((STORE (K),A(K)),K=1,I)

203 FORMAT (1X,13,2X,E21.12)

7 CONTINUE

9 WRITE (2,204)

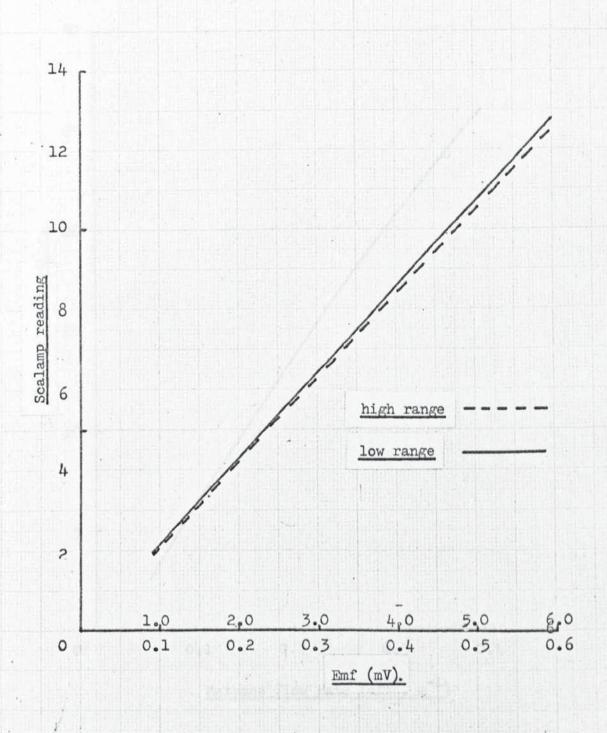
204 FORMAT (///,15H END OF PROGRAM)

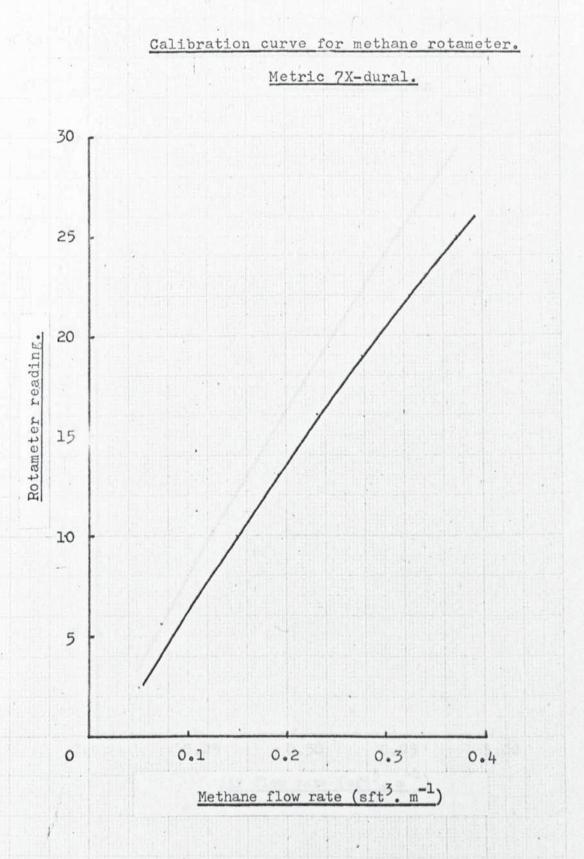
STOP
END
```

Appendix 3 Calibration data.

Figure 80.

Calibration curves for scalamp.





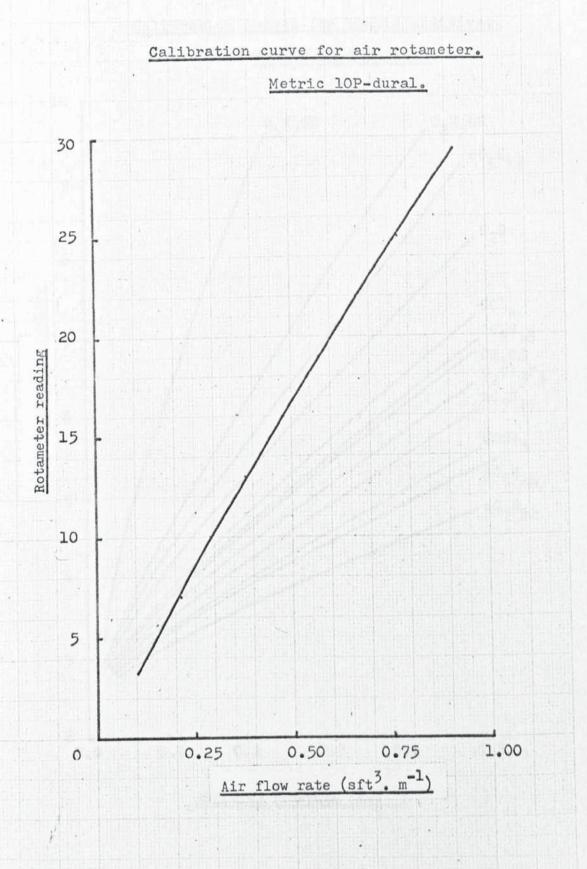
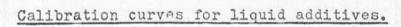
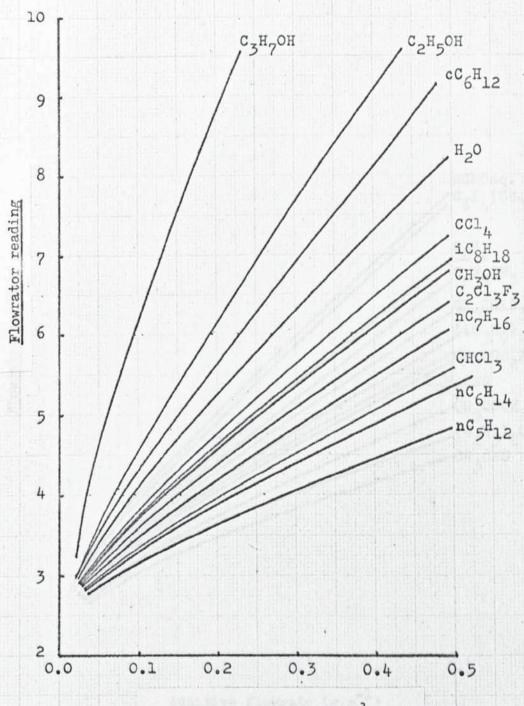


Figure 83.



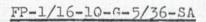
FP1/16-10-G-5/36-SA



Additive flowrate (g.m-1)

Figure 84.

# Calibration curves for liquid additives.



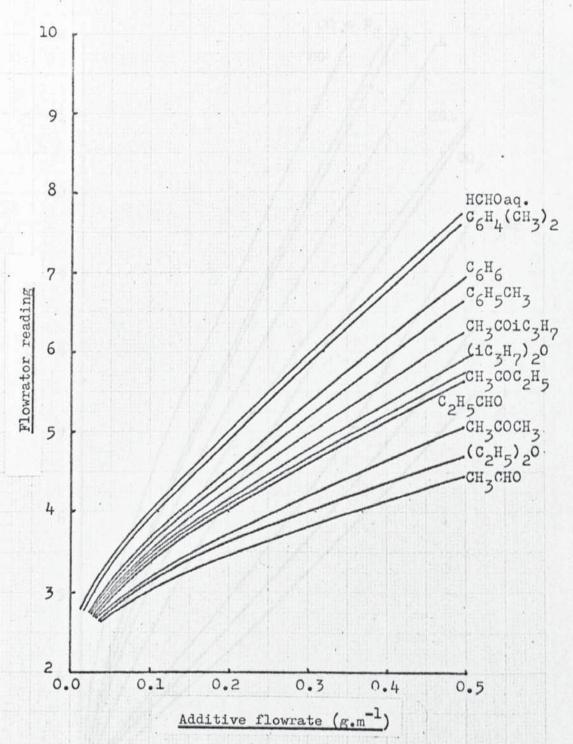


Figure 85.

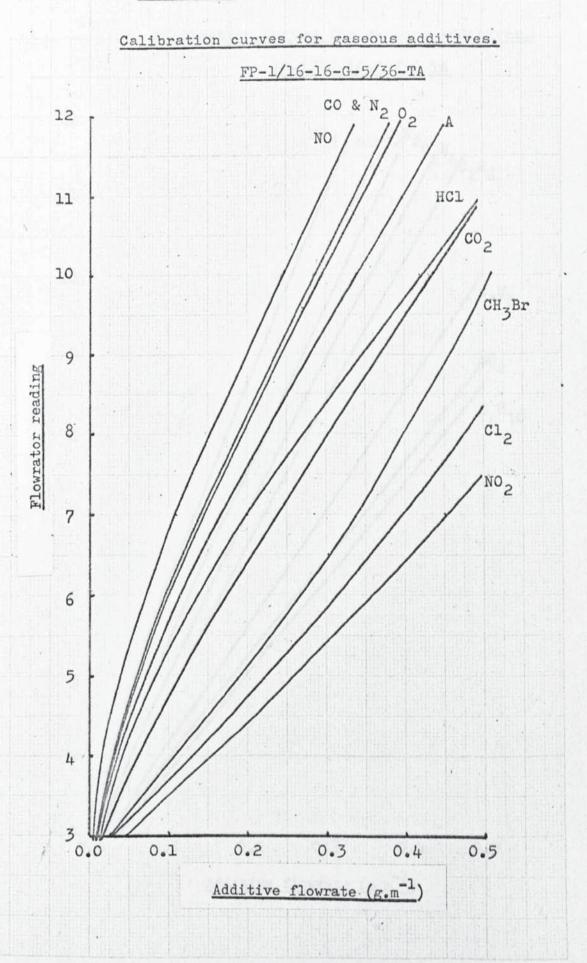


Figure 86.

### Calibration curves for gaseous additives.

## FP-1/16-16-G5/36-TA

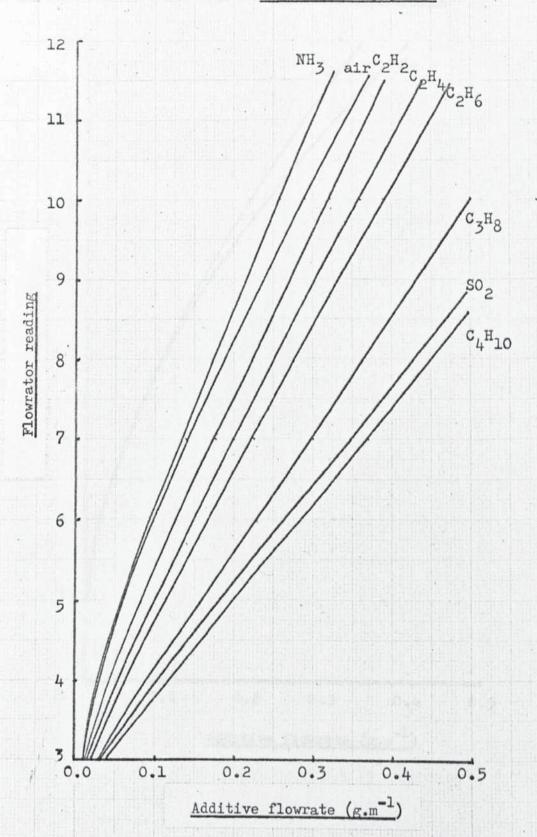


Figure 87.

3

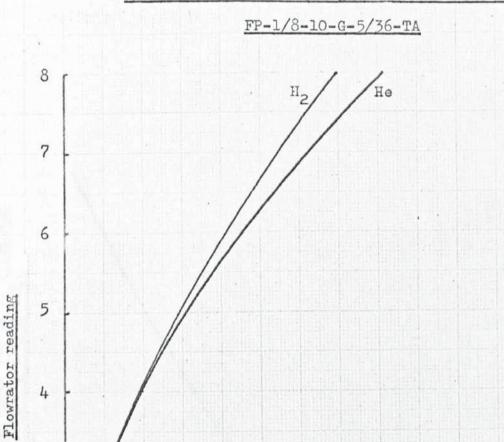
2

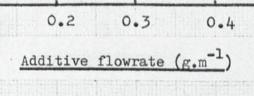
1

0

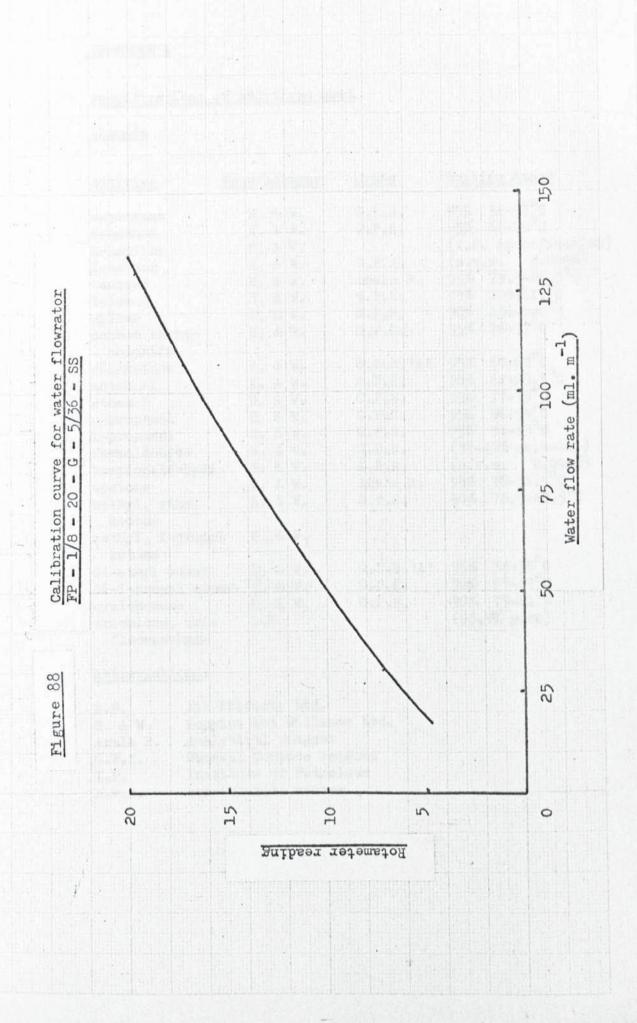
0.1

## Calibration curves for gaseous additives.





0.5



# Appendix 4

## Specifications of additives used.

### Liquids

Additive	Manufacturer	Grade	Boiling Range
n-pentane n-hexane	H. & W. H. & W.	G.P.R. G.P.R.	95% 34-37°C 95% 68-70°C
n-heptane	H. & W.		(I.P. specification)
i-octane	H. & W.	G.P.R.	(n.v.m. 0.005%)
benzene	H. & W.	Anala R.	95% 79.5-80,5°C
toluene	H. & W.	G.P.R.	95% 109 <b>-</b> 111°C
xylene	H. & W.	G.P.R.	90% 136-140°C
carbon tetra-	H. & W.	G.P.R.	95% 76 <b>-</b> 77°C
chloride			
chloroform	H. & W.	G.P.R. A	95% 60 <b>-</b> 62°C
methanol	H. & W.	G.P.R.	90% 64-65,5°C
ethanol	H. & W.	G.P.R.	95% 77 <b>-</b> 79°C
n-propanol	H. & W.	G.P.R.	95% 96-98°C
i-propanol	H. & W.	G.P.R.	95% 81-83°C
formaldehyde	H. & W.	G.P.R.	(37-41% aq.soln.)
propionaldehyde	H. & W.	G.P.R.	$(n_{\bullet}v_{\bullet}m_{\bullet} 0.004\%)$
acetone	H. & W.	Anala R.	95% 56-56.5°C
methyl, ethyl	H. & W.	G.P.R.	90% 78.5-80.5°C
ketone			
methyl, i-propyl ketone	. H. & W.		
di-ethyl ether	H. & W.	G.P.R.'A'	95% 34 <b>-</b> 35°C
di-i-propyl ethe	r H. & W.	G.P.R.	90% 67-69°C
cyclohexane	H. & W.	G.P.R.	90% 67-69°C 90% 79-81°C
trichloro, tri-	A.P.		(99.9% pure)
fluoroethane			•

## Abbreviations:

A.P.	Air Products Ltd.
H. & W.	Hopkins and Williams Ltd
Anala R.	Analytical reagent
G.P.R.	General purpose reagent
I.P.	Institute of Petroleum
n.v.m.	non-volatile matter

# Specification of additives used (continued)

### Gases

Additive	Manufacturer	Grade	Purity (%)
ethane	B.O.C.	C.P.	99.0
propane	B.O.C.	Inst.	99.5
commercial propane	Bottogas		
butane	B.O.C.	Inst.	99.5
ethylene	A.P.	C.P.	99
acetylene	B.O.C.		99.5-99.8
methyl bromide	A.P.	C.P.	99.5
hydrogen chloride	B.O.C.	Tech.	99
carbon dioxide	A.P.	H.P.	99.995
carbon monoxide	A.P.	C.P.	99.5
oxygen	B.O.C.	C.P.	99.5
hydrogen	B.O.C.	C.P.	99.9
chlorine	A.P.	H.P.	99•5
nitric oxide	A.P.	C.P.	99
sulphur dioxide	A.P.	C.P.	99.9
nitrogen dioxide	A.P.	C.P.	99.5
helium	A.P.	H.P.	99.995
argon	A.P.	H.P.	99.998
nitrogen .	B.O.C.	C.P.	99.5
ammonia	A.P.	Tech.	99.99

## Abbreviations:

A.P.	Air Products Ltd.
B.O.C.	British Oxygen Company Ltd.
C.P.	Commercial purity
H.P.	High purity
Inst.	Instrument grade
Tech.	Technical grade

Appendix 5. Results of preliminary experiments.

Table 5. Observed results for identical methaneair flames (section 8.2).

Height above burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}})$
2	3.21	6.20	0.022	1003	906
4	4.86	8.89	0.026	1067	1302
6	8.06	21.44	0.033	1160	2161
8	21.51	22.81	0.040	1323	4030
10	22.03	23.48	0.044	1388	5200
12	21.72	23.08	0.036	1401	4389
14	13.06	22.27	0.032	1313	3144
16	28.02	21.44	0.026	1235	2088
2	3.34	6.41	0.021	1026	929
4	5.15	9.43	0.026	1085	1373
6	8.07	21.44	0.031	1180	2145
8	21.56	22.87	0.042	1316	4157
10	22.19	23.67	0.043	1424	5537
12	21.74	23.10	0.038	1385	4458
14	12.09	22.17	0.030	1318	2985
16	7.61	21.34	0.025	1229	1976
2	3.50	6.65	0.023	1012	974
4	5.70	10.55	0.028	1095	1523
6	7.81	21.38	0.032	1159	2089
8	21.37	22.63	0.041	1287	3770
10	21.93	23.35	0.040	1404	4912
12	21.82	23.19	0.042	1364	4683
14	13.78	22.33	0.033	1312	3234
16	9.61	21.78	0.029	1261	2493
2	3.44	6.55	0.022	1020	955
4	4.82	8.82	0.025	1076	1286
6 .	7.05	21.19	0.029	1155	1872
8	21.56	22.87	0.043	1308	4171
10	21.90	23.30	0.041	1388	4845
12	21.62	22.95	0.038	1362	4216
14	21.28	22.51	0.033	1343	3494
16	8.61	21.57	0.025	1273	2217

Table 5 (continued)

Height above burner (cm)	Pyrome: 1 flame	ter Output e 2 flames	Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{\text{m}^{-2}\text{h}}{\text{-1}}}$
2	3.63	6.86	0.022	1034	997
4	4.28	7.90	0.022	1079	1141
6	7.43	21.29	0.029	1173	1969
8	21.46	22.75	0.042	1297	3962
10	22.09	23.55	0.045	1390	5344
12	21.53	22.84	0.036	1364	4014
14	11.05	22.03	0.029	1305	2792
16	8.03	21.44	0.023	1276	2056
2	3.10	6.04	0.021	1008	878
4	5.32	9.76	0.027	1084	1422
6	7.56	21.32	0.031	1158	2018
8	21.43	22.70	0.042	1290	3892
10	22.00	23.43	0.044	1381	5113
12	21.75	23.12	0.038	1388	4491
14	21.15	22.36	0.031	1339	3250
16	8.21	21.48	0.024	1270	2112
2	3.09	6.02	0.022	994	880
4	5.10	9.34	0.026	1082	1361
6	7.17	21,22	0.030	1150	1910
8	21.36	22.62	0.040	1294	3745
10	22.10	23.57	0.042	1418	5332
12	21.70	23.05	0.040	1360	4416
14	21.14	22.34	0.029	1360	3202
16	9.39	21.73	0.028	1265	2433
٠2	3.79	7.09	0.024	1020	1042
4	4.56	8.37	0.025	1060	1226
6	8.00	21.43	0.031	1177	2128
8	21.48	22.76	0.039	1325	3949
10	22.05	23.49	0.044	1390	5225
12	21.77	23.14	0.041	1364	4571
14	12.09	22.17	0.028	1342	2957
16	8.67	21.58	0.029	1226	2273

Table 5 (continued)

Height above burner (cm)	Pyromet 1 flame	Name and Address of the Owner, where the Party of the Owner, where the Party of the Owner, where the Party of the Owner, where the Owner, which is the Owner, where the Owner, which is the Owner, where the Owner, which is the Owner, wh	Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{m^2h.}{1}}$
2 4	3.23	6.24	0.021	1018	906
4	4.84	8.85	0.027	1055	1304
6	7.87	21.40	0.030	1182	2088
8	21.55	22.85	0.041	1321	4110
10	22.14	23.62	0.043	1416	5433
12	21.75	23.12	0.041	1361	4538
14	21.21	22.44	0.032	1341	3372
16	7.96	21.42	0.026	1232	2071
2	3.33	6.39	0.022	1012	931
4	4.63	8.48	0.025	1064	1240
6	7.59	21.33	0.029	1180	2007
8	21.62	22.94	0.043	1318	4278
10	22.21	23.71	0.045	1412	5632
12	21.65	22.99	0.035	1398	4237
14	21.22	22.45	0.033	1332	3400
16	7.85	21.40	0.025	1240	2034
2	3.29	6.32	0.024	985	933
2 4	5.27	9.65	0.029	1060	1422
6	8.76	21.60	0.033	1188	2335
8	21.46	22.75	0.042	1297	3962
10	21.98	23.40	0.041	1403	5023
12	21.73	23.09	0.037	1394	4436
14	11.64	22.11	0.030	1308	2910
16	8.32	21.51	0.025	1261	2149
4,5207	500 SERVICES	POLITER NO. 25 ST. /			
2	2.95	5.83	0.021	997	848
4	4.74	8.68	0.026	1060	1275
6	7.17	21.22	0.030	1150	1910
8	21.37	22.63	0.041	1287	3770
10	22.00	23.43	0.044	1381	5113
12	21.72	23.07	0.038	1381	4416
14	11.01	22.02	0.028	1316	2772
16	8.23	21.49	0.026	1244	2138

Table 6. Mean emissivities and 95% confidence
limits of emissivities of themethaneair flame.

<b>≤</b> E	<u>≰∈²</u>	Mean emissivity
0.265	0.005865	0.0221
0.314	0.008238	0.0262
0.368	0.011308	0.0307
0.496	0.020518	0.0413
0.516	0.022218	0.0430
0.460	0.017688	0.0383
0.368	0.011326	0.0307
0.311	0.008099	0.0259
	0.265 0.314 0.368 0.496 0.516 0.460 0.368	0.265 0.005865 0.314 0.008238 0.368 0.011308 0.496 0.020518 0.516 0.022218 0.460 0.017688 0.368 0.011326

Height	Standard	Standard	95% confidence	limits
above burner (cm)	deviation	error	Lower	Upper
2	0.0011	0.0007	0.0214	0.0228
4	0.0014	0.0009	0.0253	0.0271
6	0.0014	0.0009	0.0298	0.0316
8	0.0012	0.0008	0.0405	0.0421
10	0.0017	0.0011	0.0419	0.0441
12	0.0022	0.0014	0.0369	0.0397
14	0.0019	0.0012	0.0295	0.0319
16	0.0019	0.0012	0.0247	0.0271

Table 7. Mean temperatures and 95% confidence
limits of temperatures of the methaneair flame.

Height above burner (cm)	≰I	≰T²	Mean temperature
(cm)			
2	12129	12261587	1011
2 4	12867	13798457	1072
6	14012	16363396	1168
8	15663	20446531	1305
10	16795	23508535	1400
12	16522	22750824	1377
14	15929	21147821	1327
16	15012	18783774	1251

Height	Standard	Standard	95% confiden	ce limits
burner (cm)	deviation	error of mean	Lower	Upper
2	14	9	1002	1020
4	13	8	1064	1080
6	14	9	1159	1177
8	15	9	1296	1314
10	15	10	1390	1410
12	16	10	1367	1387
14	18	11	1316	1338
16	18	12	1239	1263

Table 8. Mean radiances and 95% confidence
limits of radiances of the methane-air flame.

Height above	≼ <u>w</u>	₹w²	Mean radiance
burner (cm)			
2	11179	10446825	932
4	15875	21118745	1323
6	24632	50753138	2053
8	47796	190700152	3983
10	62719	328451063	5227
12	52865	233243369	4405
14	37512	117898638	3126
16	26040	56783598	2170

Height	Standard	Standard	95%	confidence	1imits
above burner (cm)	deviation	error of mean	* Secondary	Lower	Upper
2	54	35		897	967
4	103	66		1257	1989
6	132	84		1969	2137
8	173	109		3874	4092
10	242	154		5073	5381
12	179	113		4292	4518
14	240	153		2973	3279
16	159	101		2069	2271

Table 9. 95% significance levels for emissivity radiance and temperature.

#### Emissivity

	Standard	95% significance	1eve1s
	error of	Lower	Upper
	difference		
2	0.0015	0.0206	0.0236
4	0.0020	0.0242	0.0282
6	0.0020	0.0287	0.0327
8	0.0017	0.0396	0.0430
10	0.0024	0.0406	0.0454
12	0.0031	0.0352	0.0414
14	0.0026	0.0281	0.0333
16	0.0026	0.0233	0.0285

#### Temperature

	Standard	95% significance	1evels
	error of difference	Upper	Lower
2	20	991	1031
4	18	1054	1090
6	20	1148	1188
8	21	1284	1326
10	21	1379	1421
12	22	1355	1399
14	25	1302	1352
16	25	1226	1276

## Radiance

	Standard	95% significance	levels
	error of difference	Upper	Lower
2	75	857	1007
4	143	1180	1466
6	184	1869	2237
8	241	3742	4224
10	338	4889	5565
12	250	4155	4655
14	334	2792	3460
16	222	1948	2392

Table 10. Observed results for identical methaneair flames with additive and mixing chamber heating (section 8.3)

Height above burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss	Flame Temp. (°C)	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{m^{-2}h.}{1}-1}$ $\underline{\text{ster.}^{-1}})$
2 4 6 8 10 12 14	3.09 4.91 7.28 21.44 22.06 21.73 21.18 7.66	6.02 8.97 21.25 22.72 23.50 23.09 22.39 21.35	0.021 0.026 0.030 0.040 0.044 0.038 0.032 0.024	1007 1070 1155 1309 1392 1384 1334	875 1313 1937 3890 5250 4448 3313 1979
2 4 6 8 10 12 14	3.47 4.52 7.16 21.41 22.16 21.66 21.32 9.31	6.60 8.29 21.22 22.68 23.63 23.00 22.57 21.71	0.022 0.025 0.029 0.042 0.044 0.036 0.034 0.030	1022 1057 1160 1287 1410 1389 1341 1239	961 1215 1899 3862 5481 4265 3582 2435
2 4 6 8 10 12 14	3.07 4.87 7.38 21.41 22.08 21.88 11.36 9.03	6.00 8.91 21.28 22.68 23.53 23.28 22.07 21.66	0.021 0.026 0.030 0.041 0.043 0.041 0.029	1006 1068 1160 1295 1404 1385 1313	873 1306 1964 3848 5280 4810 2849 2340
2 4 6 8 10 12 14	3.31 5.36 7.83 21.48 21.98 21.70 13.80 8.84	6.35 9.83 21.39 22.76 23.40 23.05 22.33 21.62	0.023 0.027 0.032 0.043 0.043 0.040 0.031 0.026	998 1086 1160 1291 1386 1360 1334 1269	932 1430 2095 3995 5057 4416 3210 2282

Table 10 (continued)

Height above burner (cm)	Pyromet 1 flame	ter Output 2 11ames	Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{-2}{\text{m}}-1}$ $\frac{\text{ster}}{}$
2	3.21	6.20	0.022	1003	906
4	5.10	9.32	0.028	1060	1373
4 6 8	8.34	21.50	0.032	1181	2221
8	21.46	22.74	0.040	1312	3920
10	21.93	23.34	0.041	1394	4916
12	21.56	22.87	0.037	1360	4085
14	12.62	22.23	0.029	1340	3048
16	7.89	21.41	0.025	1242	2045
2	3.41	6.51	0.022	1018	949
4	4.93	9.02	0.025	1083	1312
6	8.53	21.55	0.033	1179	2278
8	21.56	22.87	0.042	1316	4157
10	22.09	23.55	0.042	1415	5294
12	21.71	23.06	0.037	1389	4383
14	11.95	22.15	0.030	1315	2962
16	8.71	21.59	0.026	1264	2253

Table 11. Derived results to show effect of heating additive supply and mixing chamber on emissivity.

Height above burner (cm)	<u> </u>	<u>ई€²</u>	Mean emissivity
2 4 6 8 10 12 14 16	0.131 0.157 0.186 0.248 0.257 0.229 0.185 0.158	0.002863 0.004115 0.005778 0.010258 0.011015 0.008759 0.005723 0.004182	0.0218 0.0262 0.0310 0.0413 0.0428 0.0382 0.0308 0.0263
Height above burner (cm)	Standard deviation	Combined standard deviation	<u>t</u>
2 4 6 8 10 12 14	0.0008 0.0012 0.0015 0.0012 0.0000 0.0019 0.0019	0.0010 0.0014 0.0015 0.0013 0.0015 0.0022 0.0020	-0.30 0.00 0.20 0.00 -0.13 -0.05 0.05 0.20

4.3

Table 12. Derived results to show effect of heating additive supply and mixing chamber on

	temperat	ures.	
Height above burner (cm)	\$ <u>T</u>	₹ <u>T</u> 2	Mean temperatures
2 4 6 8 10 12 14 16	6054 6424 6995 7810 8401 8267 7977 7523	6108906 6878658 8155627 10166756 11763437 11391523 10606187 943463	1009 1071 1166 1302 1400 1378 1330 1254
Height above burner (cm)	Standard deviation	Combined standard deviation	<u>t</u>
2 4 6 8 10 12 14	9 12 11 12 11 14 12	13 13 13 14 14 15	-0.15 -0.08 -0.15 -0.21 0.0 0.07

0.18

Table 13. Derived results to show effect of heating additive supply and mixing chamber on radiance.

Height	W	_ <sub>W</sub> <sup>2</sup>	Mean radiance
burner (cm)			
2	5496	5041336	916
4	7949	10557203	1324
	12394	25726616	2066
6 8	23672	93461322	3945
10	31278	163249002	5213
12	26407	116509999	4401
14	18964	60291342	3160
16	13334	29786824	2222

Height above burner (cm)	Standard deviation	Combined standard deviation	Ē
2	37	50	-0.32
4	72	95	0.01
6	158	141	0.092
8	116	157	-3.8
10	198	230	-0.06
12	240	200	-0.02
14	266	249	0.14
16	176	164	0.32

Table 14. Observed results for single flame
with 20% acetone to evaluate
efficiency of mixing chamber (section 8.4)

First f	lame			
Height above burner (cm)	Pyro.	Pyro. output	Pyro.	Pyro.
2 4 6 8 10 12 14	3.30 4.89 7.70 1.46 2.10 1.85 1.17 8.39	3.24 5.00 7.96 1.51 2.18 1.77 1.20 7.94	3.35 5.16 8.02 1.54 2.20 1.75 1.12 8.39	3.16 4.81 7.91 1.69 2.00 1.81 1.06 8.44
Height above burner (cm)	Pyro.	Pyro.	Pyro. output	Pyro. output
2 4 6 8 10 12 14 16	3.04 4.92 7.70 1.53 2.06 1.64 1.16 8.62	3.11 4.77 7.53 1.55 2.04 1.79 1.29 8.71	3.02 4.72 7.61 1.49 2.06 1.79 1.31 8.17	3.40 4.99 7.76 1.58 2.18 1.69 1.20 8.54
Height above burner (cm)	Pyro.	Pyro.	Pyro.	Pyro.
2 4 6 8 10 12 14 16	3.33 5.00 7.85 1.47 2.22 1.63 1.14 8.62	3.11 5.16 7.67 1.45 2.20 1.85 1.09 8.25	3.26 5.03 7.95 1.63 2.07 1.88 1.09 8.19	3.27 4.80 7.83 1.55 2.20 1.70 1.12 8.44

Table 14 (continued)

## Second flame

Height above burner (cm)	Pyro.	Pyro.	Pyro.	Pyro.
2 4 6 8 10 12 14 16	3.25 5.06 7.90 1.50 2.06 1.80 1.25 7.88	3.36 4.96 7.88 1.46 2.10 1.71 1.10 8.39	3.20 4.88 7.70 1.52 2.15 1.60 1.08 8.46	3.11 4.81 7.54 1.60 2.17 1.75 1.07 8.55
Height above burner (cm)	Pyro.	Pyro.	Pyro.	Pyro. output
2 4 6 8 10 12 14 16	3.08 4.83 7.60 1.65 2.13 1.84 1.10 8.30	3.17 4.77 7.60 1.63 2.18 1.82 1.29 8.75	3.29 4.90 7.65 1.55 2.26 1.77 1.25 8.33	3.38 5.06 7.60 1.53 2.23 1.87 1.18 8.05
Height above burner (cm)	Pyro.	Pyro.	Pyro.	Pyro.
2 4 6 8 10 12 14	3.40 5.05 7.66 1.56 2.18 1.91 1.09 8.55	3.20 5.21 7.88 1.44 2.07 1.86 1.14 8.64	3.10 4.90 7.99 1.40 2.07 1.71 1.05 8.69	3.08 4.90 7.92 1.57 2.18 1.77 1.23 8.34

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Table 15. Derived results to evaluate efficiency of mixing chamber (section 8.4).

Height	Standard	Difference	<u>t</u>
above	deviation	of means	_
burner	With the William Community	Aron-east in the entermit	
(cm)			
2	0.1214	-0.00250	-0.0504
100			
4	0.1367	-0.00666	-0.1196
6	0.1577	0.04750	0.7380
8	0.07328	0.00334	0.1118
10	0.07141	-0.02250	-0.7710
12	0.08390	-0.02166	-0.6590
14	0.08062	0.01166	0.3540
16	0.2402	-0.01917	-0.1951

#### Appendix 6

### Results - observed and calculated

The results are tabulated in order of Run No. and have been divided into two parts. The smoke point and calorimeter results are presented before the pyrometer results.

Run No. L1		Ac	lditive	n-per	tane
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of	12.4 0.173	9.4 0.132	6.9 0.098	5.9 0.085	5.5 0.079
stoich.flow	0.692	0.521	0.391	0.343	0.314
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	24 53	24 51	24 50	24 48	24 47
(cal.s <sup>-1</sup> )	55.6	51.7	49.8	46.0	44.1
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	24 68	24 65	1000	24 61	24 59
(cal.s <sup>-1</sup> )	84.3	78.6	74.7	70.9	67.1
Run No. L2		Ac	lditive	n-hex	ane
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of	12.6				
stoich.flow	0.701	0.531	0.400	0.354	0.319
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 54	2 2 5 4	2 2 5 2	22 49	22 48
(cal.s <sup>-1</sup> )	61.3	61.3	57.5	51.7	49.8
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 72	22 70	22 66	22 62	22 59
(cal. s <sup>-1</sup> )	95.8	92.0	84.3	76.6	70.9

Additive cyclohe
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Run	No.	L3

Hall Hot Ho			AGT LIVE	e cycre	, ii c ii c ii c
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> ) Fraction of	7.6 0.108	7.2 0.102	6.5 0.093	5.8 0.584	5.4 0.079
stoich.flow	0.428	0.407	0.370	0.334	0.314
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 55	23 54			23 49
(cal. s <sup>-1</sup> )	61.3	59.4	57.5	53.6	49.8
2 flames					
Inlet Temp. (°C)			23 63	23 59	23 59
Outlet Temp. (°C) Heat Transfer	00	00	03	29	39
(cal. s <sup>-1</sup> )	82.4	82.4	76.6	69.0	69.0
Run No. L4		A	ditive	e chlo	roform
Run No. L4 Add. Conc.	20%	Ac	dditive	chlor 5%	roform 2.5%
8.0	20%			<del>11.</del> 13.53	700 Sept.
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s)	7.4	15% 7.1		5% 5.8	2.5%
Add. Conc. Smoke Point	7.4 0.105	7.1 0.101	10%	5.8 0.084	5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter  Air Flow (1.s )  Fraction of	7.4 0.105	7.1 0.101	10% 6.7 0.096	5.8 0.084	5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s 1)  Fraction of stoich.flow	7.4 0.105	7.1 0.101	10% 6.7 0.096	5.8 0.084	5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C)	7.4 0.105 0.417	7.1 0.101 0.401	10% 6.7 0.096 0.381	5.8 0.084 0.334	2.5% 5.4 0.079 0.314
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	7.4 0.105 0.417	7.1 0.101 0.401	6.7 0.096 0.381	5.8 0.084 0.334	5.4 0.079 0.314
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C)	7.4 0.105 0.417	7.1 0.101 0.401	10% 6.7 0.096 0.381	5.8 0.084 0.334	2.5% 5.4 0.079 0.314
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal. s 1) 2 flames	7.4 0.105 0.417 21 48 51.7	7.1 0.101 0.401	10% 6.7 0.096 0.381	5.8 0.084 0.334	2.5% 5.4 0.079 0.314
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal. s 1)  2 flames Inlet Temp. (°C)	7.4 0.105 0.417 21 48 51.7	7.1 0.101 0.401 21 49 53.6	10% 6.7 0.096 0.381 21 47 49.8	5.8 0.084 0.334 21 47 49.8	2.5% 5.4 0.079 0.314 21 46 47.9
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal. s 1) 2 flames	7.4 0.105 0.417 21 48 51.7	7.1 0.101 0.401 21 49 53.6	10% 6.7 0.096 0.381 21 47	5.8 0.084 0.334 21 47 49.8	2.5% 5.4 0.079 0.314 21 46 47.9

Run No. L5		Ad	lditiv	e aceto	ne
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s -1) Fraction of	5.4 0.079	5.3 0.078	5.2 0.078	5.1 0.075	5.1 0.075
stoich.flow	0.314	0.309	0.304	0.299	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 47		21 47		21 45
(cal.s <sup>-1</sup> )	49.8	51.7	49.8	49.8	46.0
2 flames			200	outsuts	NE-35
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 60	21 60	21 59	21 57	21 57
(cal.s <sup>-1</sup> )	74.7	74.7	72.8	69.0	69.0
Run No. L6		Ac	lditiv	e metha	nol
Run No. L6 Add. Conc.	20%	Ac	iditiv	e metha	2.5%
	20%				
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )	4.3	15%	10%		2.5% 4.8
Add. Conc.	4.3	15% 4.5 0.068	10% 4.5 0.070	<u>5%</u>	4.8 0.071
Add. Conc.  Smoke Point  Air Rotameter  Air Flow (1.s-1)  Fraction of	4.3	15% 4.5 0.068	10% 4.5 0.070	5% 4.7 0.070	4.8 0.071
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s 1)  Fraction of stoich.flow	4.3	15% 4.5 0.068	10% 4.5 0.070	5% 4.7 0.070	4.8 0.071
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s 1)  Fraction of stoich.flow  Calorimeter	4.3	15% 4.5 0.068	10% 4.5 0.070 0.268	5% 4.7 0.070	4.8 0.071
Add. Conc.  Smoke Point  Air Rotameter -1  Air Flow (1.s -1)  Fraction of stoich.flow  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C)	4.3 0.065 0.258	15% 4.5 0.068 0.268	10% 4.5 0.070 0.268	5% 4.7 0.070 0.278	2.5% 4.8 0.071 0.284
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s-1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s-1) 2 flames	4.3 0.065 0.258	15% 4.5 0.068 0.268	10% 4.5 0.070 0.268	5% 4.7 0.070 0.278	2.5% 4.8 0.071 0.284
Add. Conc.  Smoke Point  Air Rotameter 1, Air Flow (1.s 1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s 1)	4.3 0.065 0.258	15% 4.5 0.068 0.268	10% 4.5 0.070 0.268	5% 4.7 0.070 0.278	2.5% 4.8 0.071 0.284

Run No. L7	Addi	tive m	ethyl (	ethyl 1	cetone
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s <sup>-1</sup> ) Fraction of				5.2 0.076	
stoich.flow	0.329	0.324	0.314	0.304	0.304
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	23 49		23 48		23 47
(cal.s <sup>-1</sup> )	49.8	49.8	47.9	47.9	46.0
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 59	23 60	23 60	23 60	23 58
(cal.s <sup>-1</sup> )	69.0	70.9	70.9	70.9	67.1
Run No. L8		Ad	lditiv	e benze	ene
Run No. L8 Add. Conc.	20%	Ad	lditive	<u>5%</u>	ene 2.5%
	20%	50 Med (215)			
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s)	11.4	15%	10% 8.0	5%	2.5%
Add. Conc.	11.4	9.9	8.0 0.113	5% 6.6	2.5% 5.5 0.080
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s 1) Fraction of	11.4	9.9	8.0 0.113	5% 6.6 0.094	2.5% 5.5 0.080
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s 1)  Fraction of stoich.flow	11.4	9.9	8.0 0.113	5% 6.6 0.094	2.5% 5.5 0.080
Add. Conc.  Smoke Point  Air Rotameter 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1,	11.4	9.9 0.138 0.550	8.0 0.113	5% 6.6 0.094	2.5% 5.5 0.080
Add. Conc.  Smoke Point  Air Rotameter -1, Air Flow (1.s -1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C)	11.4 0.159 0.632	9.9 0.138 0.550	10% 8.0 0.113 0.448	5% 6.6 0.094 0.375	2.5% 5.5 0.080 0.319
Add. Conc.  Smoke Point  Air Rotameter 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1,	11.4 0.159 0.632 22 51 55.6	9.9 0.138 0.550	8.0 0.113 0.448	5% 6.6 0.094 0.375	2.5% 5.5 0.080 0.319
Add. Conc.  Smoke Point  Air Rotameter 1, 1, 1, 1, 2, 2, 3, 4, 2, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4,	11.4 0.159 0.632	9.9 0.138 0.550 22 51	8.0 0.113 0.448	5% 6.6 0.094 0.375	2.5% 5.5 0.080 0.319

Run No. L9	Addit	ve car	bon te	trachl	oride
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s) Fraction of	12.5			6.7 0.096	
stoich.flow	0.694	0.561	0.459	0.381	0.309
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer			100	22 48	22 48
(cal.s <sup>-1</sup> )	53.6	51.7	49.8	49.8	49.8
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer				22 62	22 62
(cal.s <sup>-1</sup> )	80.5	80.5	76.6	76.6	76.6
Run No. L10		Ad	lditive	i-pro	pano1
Run No. L10 Add. Conc.	20%	<u>Ad</u>	lditive	i-pro 5%	2.5%
	20%	9.000000			
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )	6.9	15%	10%		2.5%
Add. Conc.	6.9	6.5	10% 6.0 0.087	<u>5%</u>	2.5% 5.3 0.078
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of	6.9	6.5	10% 6.0 0.087	5.7 0.083	2.5% 5.3 0.078
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s 1) Fraction of stoich.flow	6.9	6.5	10% 6.0 0.087	5.7 0.083	2.5% 5.3 0.078
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s -1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C)	6.9	6.5 0.093 0.370	10% 6.0 0.087 0.344	5.7 0.083	2.5% 5.3 0.078
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C)	6.9 0.098 0.391	15% 6.5 0.093 0.370	10% 6.0 0.087 0.344	5% 5.7 0.083 0.329	2.5% 5.3 0.078 0.309
Add. Conc.  Smoke Point  Air Rotameter 1, 1, 1, 2, 2, 3, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4,	6.9 0.098 0.391	15% 6.5 0.093 0.370	10% 6.0 0.087 0.344	5% 5.7 0.083 0.329	2.5% 5.3 0.078 0.309
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s-1)	6.9 0.098 0.391 21 50	15% 6.5 0.093 0.370 21 50 55.6	10% 6.0 0.087 0.344 21 49 53.6	5% 5.7 0.083 0.329	2.5% 5.3 0.078 0.309 21 48 51.7

Run No. L11		Ad	ditive	n-pro	pano1
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s) Fraction of	6.9 0.098		6.0 0.087		
stoich.flow	0.391	0.375	0.344	0.334	0.309
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer			22 47		22 46
(cal.s <sup>-1</sup> )	51.7	51.7	47.9	47.9	46.0
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 67		22 63	22 63	
(cal.s <sup>-1</sup> )	86.2	82.4	78.6	78.6	74.7
Run No. L12		Addit	<u>ive</u> di	ethy1	ether
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s -1) Fraction of			6.2 0.089		
stoich.flow	0.370	0.370	0.355	0.329	0.304
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer		22 49	22 49	22 48	22 47
(cal.s <sup>-1</sup> )	55.6	51.7	51.7	49.8	47.9
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 63	22 62	22 62	22 60	22 58
(cal.s <sup>-1</sup> )	78.6	76.6	76.6	72.8	69.0

Run No. L13			Ac	ditiv	e tolue	ene
Ad	d. Conc.	20%	15%	10%	5%	2.5%
Smoke Point						
Air Rotamete Air Flow (1. Fraction of	r-1)	12.0				
stoich.flow		0.666	0.561	0.464	0.391	0.324
Calorimeter						
1 flame						
Inlet Temp. Outlet Temp. Heat Transfe		22 50	22 50		22 49	22 47
(cal.s <sup>-1</sup> )		53.6	53.6	53.6	51.7	47.9
2 flames	_					
Inlet Temp. Outlet Temp. Heat Transfer	(°C)	22 65	22 63	22 63	22 61	22 58
(ca1.s <sup>-1</sup> )		82.4	78.6	78.6	74.7	69.0
Run No. L14			Ad	lditive	i-oct	ane
Ado	d. Conc.	20%	15%	10%	5%	2.5%
Smoke Point						
Air Rotameter Air Flow (1.8 Fraction of	[-1 <sub>)</sub>	13.4 1				
stoich.flow		0.744	0.566	0.412	0.365	0.319
Calorimeter						
1 flame						
Inlet Temp. Outlet Temp. Heat Transfer	(°C)	23 49	23 51	23 50		23 48
(cal.s <sup>-1</sup> )		49.8	53.6	51.7	47.9	47.9
2 flames						
Inlet Temp. ( Outlet Temp. ( Heat Transfer	(°c)	23 66		23 62	23 60	23 57
(cal.s <sup>-1</sup> )		82.4	80.5	74.7	70.9	65.1

Run No. L15	dditive	methy	l i-pro	pyl ke	etone
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s 1) Fraction of		6.4 0.092			
stoich.flow	0.370	0.365	0.344	0.324	0.309
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 48	22 48		22 47	22 45
(cal.s <sup>-1</sup> )	49.8	49.8	49.8	47.9	44.1
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C)	22 60	22 60	22 59	22 57	22 56
Heat Transfer			,,	31	30
(cal.s <sup>-1</sup> )	72.8	72.8	70.9	67.1	65.1
Run No. L16	28	Ac	lditive	water	r .
Add. Conc.	20%	15%	10%	5%	2.5%
Air Rotameter Air Flow (1.s-1) Fraction of	incomp: vapori			4.8 0.071	4.9 0.073
stoich.flow				0.284	0.289
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer				22 42	2 2 4 2
(cal.s <sup>-1</sup> )				38.3	38.3
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	¥			22 56	22 56
(cal.s <sup>-1</sup> )				65.1	65.1

		er s			# 5744
Run No. L17		Ad	lditive	n-hep	tane
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s 1) Fraction of	12.9 1				
stoich.flow	0.716	0.556	0.438	0.365	0.319
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 52	22 50	22 49	22 49	22 47
(cal.s <sup>-1</sup> )	57.5	53.6	51.7	51.7	47.9
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 70	22 67	22 64	22 61	22 58
(cal.s <sup>-1</sup> )	92.0	86.2	80.5	74.7	69.0
Run No. L18 Addi	tive wa	ter +	0.005M	surfa	ctant
					2.5%
Add. Conc.	20%	15%	10%	<u>5%</u>	2.3%
Smoke Point					
Air Rotameter 1 Air Flow (1.s) Fraction of				4.9 0.073	
stoich.flow	0.278	0.284	0.289	0.289	0.294
Calorimeter					
1 flame Inlet Temp. (°C) Outlet Temp. (°C)	23 44	23 44		23 42	23 42
Heat Transfer		TORISON :	(6) <del>(5)</del>	SANTA .	
(cal.s <sup>-1</sup> )	40.2	40.2	38.3	36.4	36.4
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 60	23 58	23 58	23 56	23 54
(221 2-1)	70.0	67 1	67 7		EO 4

67.1 67.1

59.4

63.2

70.9

(cal.s<sup>-1</sup>)

Run No. L19 Add	itive w	ater +	0.01M	surfac	tant
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s 1) Fraction of				4.9 0.073	
stoich.flow	0.284	0.284	0.284	0.289	0.294
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 45		22 43	2 2 4 2	22 42
(cal. s <sup>-1</sup> )	44.1	44.1	40.2	38.3	38.3
2 flames Inlet Temp. (°C)	22	22	22	22	22
Outlet Temp. (°C) Heat Transfer	59	58	55	55	54
(cal. s <sup>-1</sup> )	70.9	69.0	63.2	63.2	61.3
Run No. L20		Ac	lditiv	<u>xyler</u>	ne
Run No. L20 Add. Conc.	20%	<u>Ad</u>	lditive	<u>xyler</u> <u>5%</u>	ne 2.5%
Add. Conc.	20%	in the second		-3	
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1)	14.0	15% 10.9	10% 7.8	<u>5%</u>	2.5% 5.6
Add. Conc.	14.0	15% 10.9 0.152	7.8 0.110	<u>5%</u>	2.5% 5.6 0.082
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of	14.0	15% 10.9 0.152	7.8 0.110	5% 6.6 0.094	2.5% 5.6 0.082
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1) Fraction of stoich.flow	14.0	15% 10.9 0.152	7.8 0.110	5% 6.6 0.094	2.5% 5.6 0.082
Add. Conc.  Smoke Point  Air Rotameter -1  Air Flow (1.s -1)  Fraction of stoich.flow  Calorimeter	14.0	15% 10.9 0.152	7.8 0.110	5% 6.6 0.094	2.5% 5.6 0.082
Add. Conc.  Smoke Point  Air Rotameter -1 Air Flow (1.s -1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C)	14.0 0.196 0.779	15% 10.9 0.152 0.605	7.8 0.110 0.438	5% 6.6 0.094 0.375	2.5% 5.6 0.082 0.325
Add. Conc.  Smoke Point  Air Rotameter -1, Air Flow (1.s -1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s -1) 2 flames	14.0 0.196 0.779	15% 10.9 0.152 0.605	7.8 0.110 0.438 21 49 53.6	5% 6.6 0.094 0.375 21 47 49.8	2.5% 5.6 0.082 0.325 21 45 46.0
Add. Conc.  Smoke Point  Air Rotameter 1, Air Flow (1.s 1)  Fraction of stoich.flow  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s 1)	14.0 0.196 0.779	15% 10.9 0.152 0.605	7.8 0.110 0.438	5% 6.6 0.094 0.375	2.5% 5.6 0.082 0.325

1 32 1

Run No. L21	Add	itive o	di-i-p	ropyl e	ther
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of			6.4 0.092		
stoich.flow	0.401	0.391	0.365	0.329	0.309
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 46	21 44			21 42
(cal.s <sup>-1</sup> )	47.9	44.1	42.2	42.2	40.2
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer		21 59		21 56	21 56
(calss <sup>-1</sup> )	74.7	72.8	69.0	67.1	67.1
Run No. L22	Add	ltive I	propion	naldely	/de
Run No. L22 Add. Conc.		no simbolini		and the second	7de 2.5%
		<u>15%</u>		en de la composition della com	
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )	20%	15% 4.6		<u>5%</u>	2.5%
Add. Conc. Smoke Point	4.5 0.068	15% 4.6 0.069	10%	5% 4.8 0.071	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )  Fraction of	4.5 0.068	15% 4.6 0.069	10% 4.6 0.069	5% 4.8 0.071	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1)  Fraction of stoich.flow  Calorimeter  1 flame	4.5 0.068	15% 4.6 0.069	10% 4.6 0.069	5% 4.8 0.071	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1)  Fraction of stoich.flow  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	20% 4.5 0.068 0.268	4.6 0.069 0.273	10% 4.6 0.069 0.273	5% 4.8 0.071 0.284	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1,	20% 4.5 0.068 0.268	15% 4.6 0.069 0.273	10% 4.6 0.069 0.273	5% 4.8 0.071 0.284 21 45	2.5% 4.9 0.073 0.289
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1)  Fraction of stoich.flow  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	20% 4.5 0.068 0.268	15% 4.6 0.069 0.273	10% 4.6 0.069 0.273	5% 4.8 0.071 0.284 21 45	2.5% 4.9 0.073 0.289
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )  Fraction of stoich.flow  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s <sup>-1</sup> )	20% 4.5 0.068 0.268	4.6 0.069 0.273 21 46 47.9	10% 4.6 0.069 0.273 21 46 47.9	5% 4.8 0.071 0.284 21 45 46.0	2.5% 4.9 0.073 0.289

Run No. L23		Ad	lditive	ethar	101
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s) Fraction of	5.5 0.080	5.4 0.079	5.3 0.078	5.1 0.075	5.0 0.074
stoich.flow	0.319	0.314	0.309	0.299	0.294
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer		21 48	21 47	21 46	21 44
(cal.s <sup>-1</sup> )	51.7	51.7	49.8	47.9	44.1
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 61	21 62	21 60	21 58	21 58
(cal.s <sup>-1</sup> )	76.6	78.6	74.7	70.9	70.9
Pup No. 124		A 4 4 4 4	ilva f	. w.m. o. 1. d. d	1 wdo
Run No. L24		2	ive fo		
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> ) Fraction of		4.2 0.064			
stoich.flow	0.248	0.253	0.263	0.273	0.284
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer		22 43	22 43	22 43	22 43
(cal.s <sup>-1</sup> )	44.1	40.2	40.2	40.2	40.2
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 61	22 59	22 58	22 58	22 58
(cal.s <sup>-1</sup> )	74.7	70.9	69.0	69.0	69.0

Run No. L25 Addi	tive tr	ichlore	-trif1	uoroet	hane
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s 1) Fraction of		4.7 0.070			
stoich.flow	0.278	0.278	0.284	0.289	0.291
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 47		21 46	21 45	21 43
(cal.s <sup>-1</sup> )	49.8	49.8	47.9	46.0	42.2
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 61	21 60	21 58	21 58	21 56
(cal.s <sup>-1</sup> )	76.6	74.7	70.9	70.9	67.1
D N 01					
Run No. G1			dditiv	e nitro	
Run No. G1 Add. Conc.	20%	A 0	dditiv	e nitro	2.5%
Add. Conc.	20%				
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s)	5.0		10%	5% 5.1	<u>2.5%</u>
Add. Conc.	5.0 0.074	15%	10% 5.1 0.075	5% 5.1 0.075	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter  Air Flow (1.s <sup>-1</sup> )  Fraction of	5.0 0.074	15% 5.0 0.074	10% 5.1 0.075	5% 5.1 0.075	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s) Fraction of stoich.flow  Calorimeter  1 flame	5.0 0.074	15% 5.0 0.074	10% 5.1 0.075	5% 5.1 0.075	2.5% 4.9 0.073
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )  Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C)	5.0 0.074 0.294	5.0 0.074 0.294	10% 5.1 0.075 0.299	5.1 0.075 0.299	2.5% 4.9 0.073 0.289
Add. Conc.  Smoke Point  Air Rotameter 1 Air Flow (1.s) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	5.0 0.074 0.294	5.0 0.074 0.294	10% 5.1 0.075 0.299	5.1 0.075 0.299	2.5% 4.9 0.073 0.289
Add. Conc.  Smoke Point  Air Rotameter -1, Air Flow (1.s) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C)	5.0 0.074 0.294	5.0 0.074 0.294	10% 5.1 0.075 0.299	5.1 0.075 0.299	2.5% 4.9 0.073 0.289
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s 1)  Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s 1)  2 flames	5.0 0.074 0.294 22 41 36.4	5.0 0.074 0.294	10% 5.1 0.075 0.299 22 43 40.2	5.1 0.075 0.299	2.5% 4.9 0.073 0.289
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s-1) Fraction of stoich.flow  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s )	5.0 0.074 0.294	5.0 0.074 0.294 22 41 36.4	10% 5.1 0.075 0.299	5.1 0.075 0.299	2.5% 4.9 0.073 0.289

Run No. G2		A	dditiv	e argo	n
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s) Fraction of	5.2 0.076	5.1 0.075	5.3 0.077	5.1 0.075	5.1 0.075
stoich.flow	0.384	0.299	0.309	0.299	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	23 40	23 40	23 42	23 42	23 44
(cal.s <sup>-1</sup> )	32.6	32.6	36.4	36.4	40.2
2 flames					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	23 54	23 56	23 57	23 58	23 60
(cal.s <sup>-1</sup> )	59.4	63.2	65.1	67.1	70.9
*					
Run No. G3	4	Additiv	ve carl	on di	oxide
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> ) Fraction of		5.2 0.076			
stoich.flow	0.309	0.304	0.304	0.304	0.299
Calanimatan					
Calorimeter			D.		
1 flame Inlet Temp. (°C)	24	0.4	0.4	0.1	, ,
Outlet Temp. (°C) Heat Transfer	47	24 47	24 46	24 46	24 45
(cal.s <sup>-1</sup> )	44.1	44.1	69.0	65.1	65.1
2 flames					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	24 61	24 61	24 60	24 58	24 58
(cal.s <sup>-1</sup> )	70.9	70.9	69.0	65.1	65.1

Run No. G4	Addit	ive co	mmerci	al pro	pane
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of				5.8 0.084	
stoich.flow	0.433	0.401	0.365	0.334	0.314
Calorimeter					
1 flame					2 2
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	2 2 4 9	22 50	22 50	22 50	22 49
(cal.s <sup>-1</sup> )	51.7	53.6	53.6	53.6	51.7
2 flames			•		
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	2 2 6 5	22 64	22 62	22 60	22 58
(cal.s <sup>-1</sup> )	82.4	80.5	76.6	72.8	69.0
Run No. G5	A	dditiv	e carb	on mone	oxide
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s 1) Fraction of	6.0 0.087	5.9 0.086	5.6 0.082	5.3 0.077	5.0 0.074
stoich.flow	0.344	0.339	0.324	0.309	0.294
Calorimeter					
1 flame	•				
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	24 43	24 44	24 44	24 45	24 45
(cal.s <sup>-1</sup> )	36.4	38.3	38.3	40.2	40.2
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C)	24	24	24	24	24 61
Heat Transfer	59	59	5 9	60	OT

Run No. G6 Additive				e oxyge	en
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s) Fraction of				4.2 0.064	
stoich.flow	0.194	0.209	0.234	0.253	0.268
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	24 47	24 48	24 47	24 47	24 46
(cal.s <sup>-1</sup> )	44.1	46.0	44.1	44.1	42.2
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	24 61	24 61	24 60	24 59	24 58
(cal.s <sup>-1</sup> )	70.9	70.9	69.0	67.1	65.1
Run No. G7		Ac	ditiv	e acet	ylene
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s 1) Fraction of				5.4 0.079	
stoich.flow	0.350	0.344	0.334	0.314	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 50	22 51	22 49	22 47	22 44
(cal.s <sup>-1</sup> )	53.6	55.6	51.7	47.9	42.2
2 flames					
Inlet Temp. (°C)					
Outlet Temp. (°C) Heat Transfer	22 67	22 66	22 63	22 60	22 57

Run No. G8 Additive methane					
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of		5.8 0.084			
stoich.flow	0.350	0.334	0.324	0.304	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 52	22 50	22 49	22 47	22 45
(cal.s <sup>-1</sup> )	57.5	53.6	51.7	47.9	44.1
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 67	22 66	22 63	22 61	22 58
(cal.s <sup>-1</sup> )	86.2	84.3	78.6	74.7	69.0
Run No. G9		<u>A</u>	dditiv	e air	
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> )	-	-	-	-	-
Air Flow (1.s <sup>-1</sup> ) Fraction of	-	-	-	_	-
stoich.flow	-	-	-	-	-
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C)	22 40	22	22 42	22 41	22
Heat Transfer	40	44	44	41	44
(cal.s <sup>-1</sup> )	34.5	30.3	38.3	36.4	38.3
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 57	22 57	22 57	22 55	22 55
(cal.s <sup>-1</sup> )	67.1	67.1	67.1	63.2	63.2

Run No. G10		Ad	lditive	hydro	gen
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s) Fraction of	3.9 0.060	4.0 0.061	4.2 0.064	4.5 0.068	4.7 0.070
stoich.flow	0.239	0.243	0.253	0.268	0.278
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 43	and the same	23 44	23 44	23 43
(cal.s <sup>-1</sup> )	38.3	38.3	40.2	40.2	38.3
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 55	23 56	23 57	23 57	23 57
(cal.s <sup>-1</sup> )	61.3	63.2	65.1	65.1	65.1
Run No. G11		A	dditiv	e heli	um
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> ) Fraction of				5.2 0.076	
stoich.flow	0.304	0.304	0.299	0.304	0.304
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 44	23 44	23 42	23 42	23 44
(cal.s <sup>-1</sup> )	40.2	40.2	36.4	36.4	40.2
2 flames					
a rranca					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 59	23 58	23 58	23 58	23 59

Run No. G	12	<u>4</u>	dditi	ze metl	yl bro	mide
	Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Poi	nt					
Air Rotam Air Flow Fraction	eter-1 (1.s <sup>-1</sup> )	12.6			7.3 0.104	
stoich.fl		0.699	0.610	0.502	0.412	0.329
Calorimet	er					
1 flame	٠٥ .					0.1
Inlet Tem Outlet Te Heat Tran	mp.( C)		7 7 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	21 49		21 46
(cal.s <sup>-1</sup> )		55.6	53.6	53.6	53.6	47.9
2 flames						na sana na m
Inlet Tem Outlet Te Heat Tran	mp.(C)	21 63	21 63	21 61	21 59	21 57
(cal.s <sup>-1</sup> )		80.5	80.5	76.6	72.8	69.0
Run No. G	13		<u>A</u>	dditiv	e etha	ne
	Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Poi	nt					
Air Rotam Air Flow Fraction	eter (1.s <sup>-1</sup> )				5.7 0.083	
stoich.fl		00.417	0.391	0.360	0.329	0.309
Calorimet	er		<b>%</b>			
1 flame	0					F-23749
Inlet Tem Outlet Te Heat Tran	mp.(°C)	22 52	22 50	22 48	22 48	22 46
(cal.s <sup>-1</sup> )		57.5	53.6	49.8	49.8	46.0
2 flames						
Inlet Tem Outlet Te Heat Tran	mp.(°C)	22 69	22 69	22 65	22 61	22 59
(cal.s <sup>-1</sup> )		90.1	90.1	82.4	74.7	70.9

Run No. G14		Ad	lditive	propa	ine
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s) Fraction of	7.6 0.108	7.0 0.100	6.4 0.092	5.7 0.083	5.3 0.078
stoich.flow	0.427	0.396	0.365	0.329	0.309
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	24 52			24 50	24 48
(cal.s <sup>-1</sup> )	53.6	53.6	51.7	49.8	46.0
2 flames					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	67	24 66	24 63	24 60	24 59
(cal.s <sup>-1</sup> )	82.4	80.5	74.7	69.0	67.1
Run No. G15		A	dditiv	e buta	ne
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s 1) Fraction of				5.8 0.084	
stoich.flow	0.448	0.412	0.370	0.334	0.314
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 51	23 49	23 48	23 45	23 44
(cal.s <sup>-1</sup> )	53.6	49.8	47.9	42.2	40.2
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 69	23 65	23 62	23 58	23 55
(cal.s <sup>-1</sup> )	88.1	80.5	74.7	67.1	61.3

Run No. G16		Ac	iditiv	ethy1	ene
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter 1 Air Flow (1.s) Fraction of				5.8 0.084	
stoich.flow	0.407	0.386	0.360	0.334	0.314
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer				22 48	22 46
(cal.s <sup>-1</sup> )	51.7	51.7	49.8	49.8	46.0
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	22 64	22 63	22 61	22 60	22 58
(cal.s <sup>-1</sup> )	80.5	78.6	74.7	72.8	69.0
Run No. G17	Add	ditive	nitro	gen dio	oxide
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> ) Fraction of				5.2 0.076	
stoich.flow	0.334	0.334	0.324	0.304	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 46	23 46	23 44	23 46	23 46
(cal.s <sup>-1</sup> )	44.1	44.1	40.2	44.1	44.1
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 62	23 60	23 59	23 61	23 61
(cal.s <sup>-1</sup> )	74.7	70.9	69.0	72.8	72.8

Run No. G18	Addi	tive h	nydroge	en chlo	ride
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter Air Flow (1.s <sup>-1</sup> ) Fraction of				5.1 0.075	
stoich.flow	0.309	0.304	0.304	0.299	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	21 44			21 46	21 46
(cal.s <sup>-1</sup> )	44.1	46.0	47.9	47.9	47.9
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	21 58	21 58	21 58	21 60	21 60
(cal.s <sup>-1</sup> )	70.9	70.9	70.9	74.7	74.7
Run No. G19		A	dditiv	e chlo	rine
Run No. G19 Add. Conc.	20%	<u>A</u>	dditiv	e chlos	rine 2.5%
	20%	-			
Add. Conc.	7.8	15%	10%		2.5% 5.4
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s <sup>-1</sup> )	7.8 0.110	7.4 0.105	10% 6.6 0.094	5% 5.8	2.5% 5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter Air Flow (1.s) Heat Transfer	7.8 0.110	7.4 0.105	10% 6.6 0.094	5% 5.8 0.084	2.5% 5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s 1)  Heat Transfer  (cal.s 1)	7.8 0.110	7.4 0.105	10% 6.6 0.094	5% 5.8 0.084	2.5% 5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter 1  Air Flow (1.s 1)  Heat Transfer (cal.s 1)  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C)	7.8 0.110	7.4 0.105	10% 6.6 0.094	5% 5.8 0.084	2.5% 5.4 0.079
Add. Conc.  Smoke Point  Air Rotameter—1 Air Flow (1.s — 1)  Heat Transfer (cal.s — 1)  Calorimeter  1 flame	7.8 0.110 0.438	7.4 0.105 0.417	10% 6.6 0.094 0.375	5.8 0.084 0.334	2.5% 5.4 0.079 0.314
Add. Conc.  Smoke Point  Air Rotameter -1, Air Flow (1.s -1) Heat Transfer (cal.s -1)  Calorimeter  1 flame Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s -1)  2 flames	7.8 0.110 0.438	7.4 0.105 0.417	10% 6.6 0.094 0.375	5.8 0.084 0.334	2.5% 5.4 0.079 0.314
Add. Conc.  Smoke Point  Air Rotameter -1, Air Flow (1.s -1) Heat Transfer (cal.s -1)  Calorimeter  1 flame  Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer (cal.s -1)	7.8 0.110 0.438	7.4 0.105 0.417	10% 6.6 0.094 0.375	5.8 0.084 0.334	2.5% 5.4 0.079 0.314

Run No. G20		Addit	ive ni	itric o	xide
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s <sup>-1</sup> ) Fraction of				5.4 0.079	
stoich.flow	0.360	0.350	0.329	0.314	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer			23 44	23 43	23 43
(cal,s <sup>-1</sup> )	42.2	42.2	40.2	38.3	38.3
2 flames					
				23 60	23 58
(cal,s <sup>-1</sup> )	70.9	70.9	72.8	70.5	67.1
Run No. G21	<u>A</u>	dditiv	e sulp	hur die	oxide
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s) Fraction of				5.1 0.075	
stoich.flow	0.294	0.294	0.294	0.299	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp. (°C)	24 44	24 45	24 45	24 46	24 46
Heat Transfer (cal.s <sup>-1</sup> )	38.3	40.2	40.2	42.2	42.2
		4002	4012	74.4	70.0
2 flames Inlet Temp. (°C)	24	24	24	24	24
Outlet Temp. (°C) Heat Transfer	58	58	60	60	61
(cal.s <sup>-1</sup> )	65.1	65.1	69.0	69.0	70.9

Run No. G22	Run No. G22 Additive ammonia				
Add. Conc.	20%	15%	10%	5%	2.5%
Smoke Point					
Air Rotameter-1 Air Flow (1.s <sup>-1</sup> ) Fraction of	5.2 0.076	5.2 0.076	5.2 0.076	5.1 0.075	5.1 0.075
stoich.flow	0.304	0.304	0.304	0.299	0.299
Calorimeter					
1 flame					
Inlet Temp (°C) Outlet Temp(°C) Heat Transfer	22 42	2 2 4 2	22 44	22 44	22 46
(cal.s <sup>-1</sup> )	38.3	38.3	42.2	42.2	46.0
2 flames					
Inlet Temp (°C) Outlet Temp(°C) Heat Transfer	22 59	22 59	22 60	2 2 6 2	64
(cal.s <sup>-1</sup> )	70.9	70.9	72.8	76.6	80.5
Run No. G23	Ad	ditive	sulph	ur tri	oxide
Add. Conc.	20%	15%	10%	<u>5%</u>	2.5%
Smoke Point					
Air Rotameter -1 Air Flow (1.s 1) Fraction of		5.2 0.076			
stoich.flow	0.304	0.304	0.299	0.299	0.299
Calorimeter					
1 flame					
Inlet Temp. (°C) Outlet Temp.(°C) Heat Transfer	23 45	23 45	23 46	23 48	23 48
(cal.s <sup>-1</sup> )	42.2	42.2	44.1	47.9	47.9
2 flames					
Inlet Temp. (°C) Outlet Temp. (°C) Heat Transfer	23 51		23 51	23 53	23 53
(cal.s <sup>-1</sup> )	53.6	53.6	53.6	57.5	57.5

Add. Conc. (%)	Ht. above Burner (cm)		r Output 2 flames	Emiss.	Flame Temp.	$\frac{\text{Rad.}}{(\text{Kcal.}}$ $\frac{-2}{\text{m}-h} \cdot -1$ ster.
20	2	2.33	4.92	0.020	961	720
	4	4.82	8.82	0.027	1054	1300
	6	21.29	22.51	0.055	1176	3761
	8	22.80	24.42	0.082	1295	7692
	10	24.07	26.13	0.104	1388	12278
	12	24.71	27.05	0.113	1436	14952
	14	23.83	25.80	0.100	1371	11336
	16	22.80	24.42	0.082	1295	7692
15	2 4 6 8 10 12 14 16	2.61 5.00 21.24 22.57 23.78 24.03 22.73 22.50	5.33 9.14 22.45 24.14 25.75 26.09 24.37 24.05	0.021 0.028 0.055 0.072 0.090 0.095 0.059	971 1054 1166 1305 1402 1416 1401 1285	780 1348 3662 6935 10999 11992 7197 6773
10	2	3.12	6.07	0.023	984	893
	4	5.54	10.18	0.030	1065	1494
	6	13.46	22.29	0.049	1177	3365
	8	22.31	23.81	0.062	1314	6106
	10	23.32	25.14	0.074	1409	9200
	12	23.13	24.89	0.070	1401	8539
	14	23.53	24.10	0.061	1356	6676
	16	21.77	23.14	0.052	1281	4711
5	2	3.52	6.67	0.024	1001	982
	4	5.58	10.27	0.030	1068	1505
	6	9.69	21.78	0.038	1176	2598
	8	21.89	23.29	0.050	1317	4964
	10	22.65	24.27	0.055	1413	6903
	12	22.26	23.77	0.048	1397	5799
	14	21.54	22.85	0.039	1338	4079
	16	10.58	21.95	0.031	1269	2723
2.5	2.2	3.76	7.04	0.024	1018	1035
	4	2.10	9.33	0.027	1071	1367
	6	8.34	21.51	0.033	1172	2232
	8	21.57	22.88	0.043	1310	4192
	10	22.22	23.71	0.046	1405	5665
	12	21.90	23.30	0.041	1388	4850
	14	21.19	22.41	0.033	1326	3349
	16	8.62	21.57	0.027	1248	2241

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometer 1 flame 2	The same of the sa	Emiss.	Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14 16	2.36 4.37 21.17 22.83 24.08 24.65 23.80 21.85	4.96 8.04 22.36 24.47 26.14 26.94 25.75 23.24	0.020 0.025 0.055 0.085 0.108 0.116 0.103 0.040	964 1048 1152 1288 1376 1418 1356 1388	726 1181 3525 7844 12386 14726 11250 4723
15	2 4 6 8 10 12 14	3.42 4.70 21.19 22.67 23.79 24.16 23.42 22.57	6.52 8.60 22.38 24.27 25.75 26.27 25.26 24.13	0.023 0.026 0.054 0.078 0.095 0.100 0.090 0.076	1006 1057 1162 1293 1384 1413 1353 1286	956 1264 3555 7287 11109 12551 9772 6971
10	2 4 6 8 10 12 14 16	2.81 5.25 11.85 22.32 23.30 23.22 22.48 21.81	5.61 9.61 22.12 23.83 25.12 25.01 24.04 23.18	0.022 0.029 0.049 0.065 0.075 0.072 0.062	973 1059 1152 1300 1402 1404 1343 1276	824 1417 3134 6185 9166 8850 8562 4832
5	2 4 6 8 10 12 14	3.70 5.48 9.40 21.84 23.17 22.28 21.53 10.36	6.96 10.06 21.73 23.21 24.94 23.79 22.83 21.91	0.025 0.030 0.039 0.050 0.075 0.049 0.040	1003 1062 1158 1307 1383 1393 1327 1263	1028 1480 2539 4836 8753 5864 4068 2678
2.5	2 4 6 8 10 12 14 16	3.68 5.13 8.15 21.54 22.32 21.87 21.24 8.06	6.92 9.39 21.46 22.84 23.84 23.26 22.47 21.44	0.025 0.028 0.033 0.043 0.048 0.042 0.033 0.026	1001 1062 1164 1303 1407 1374 1336	1023 1381 2182 4117 5934 4798 3431 2095

Add. Conc. (%)	Ht.  above Burner (cm)		r Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}})$
20	2 4 6 8 10 12 14	2.07 4.12 8.58 22.79 23.63 24.12 23.41 22.49	4.56 8.36 21.56 24.42 25.55 26.23 25.24 24.04	0.020 0.026 0.034 0.075 0.084 0.090 0.084 0.070	940 1048 1172 1324 1407 1447 1376 1302	673 1321 2299 7582 10384 12224 9634 6688
15	2	2.36	4.96	0.021	951	731
	4	4.65	8.51	0.026	1054	1252
	6	21.23	22.44	0.054	1170	3635
	8	22.46	24.00	0.065	1322	6532
	10	23.48	25.35	0.077	1417	9757
	12	23.84	25.84	0.082	1444	11075
	14	22.89	24.57	0.070	1365	7828
	16	22.01	23.42	0.059	1280	5334
10	2	3.09	6.03	0.024	971	892
	4	5.12	9.36	0.028	1061	1378
	6	12.54	22.20	0.047	1177	3228
	8	22.16	23.62	0.057	1318	5671
	10	23.02	24.75	0.066	1406	8143
	12	22.89	24.58	0.062	1409	7708
	14	22.14	23.60	0.053	1340	5576
	16	21.38	22.64	0.044	1265	3825
5	2	3.44	6.55	0.025	984	970
	4	5.38	9.86	0.029	1066	1448
	6	9.36	21.72	0.037	1173	2513
	8	21.70	23.04	0.045	1317	4468
	10	22.40	23.94	0.050	1406	6169
	12	21.99	23.42	0.042	1396	5065
	14	21.40	22.67	0.036	1338	3765
	16	9.78	21.81	0.028	1279	2521
2.5	2 4 6 8 10 12 14 16	3.52 5.10 8.32 21.54 22.20 22.01 21.54 8.41	6.67 9.33 21.50 22.84 23.69 23.45 22.84 21.53	0.024 0.027 0.033 0.042 0.046 0.044 0.043	1001 1071 1171 1312 1402 1384 1303 1252	982 1367 2227 4111 5622 5145 4117 2181

Add. Conc. (%)	Ht. above Burner (cm)		r Output 2 flames	Emiss.	Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	3.07 5.14 21.20 22.60 23.21 23.18 22.83 21.81	5.99 9.40 22.39 24.17 24.98 24.94 24.49	0.023 0.029 0.056 0.075 0.083 0.081 0.072	980 1052 1152 1295 1352 1356 1346 1241	882 1390 3589 7036 8977 8847 7680 4897
15	2 4 6 8 10 12 14 16	3.37 5.65 21.23 22.42 23.24 23.11 22.73 21.77	6.44 10.41 22.45 23.95 25.02 24.86 24.36 23.12	0.024 0.031 0.034 0.069 0.078 0.073 0.066 0.055	991 1062 1323 1296 1378 1383 1360 1263	951 1529 3423 6486 8998 8520 7294 4751
10	2 4 6 8 10 12 14	3.70 5.90 11.06 22.13 23.16 22.91 22.08 21.39	6.96 10.97 22.01 23.58 24.93 24.61 23.53 22.65	0.025 0.031 0.046 0.058 0.071 0.063 0.052 0.043	1003 1076 1154 1307 1400 1406 1337 1275	1028 1592 2962 5609 8644 7773 5427 3832
5	2 4 6 8 10 12 14 16	3.70 5.39 10.06 21.86 23.18 22.51 21.63 10.58	6.96 9.88 21.85 23.24 24.95 24.10 22.96 21.95	0.025 0.029 0.040 0.049 0.079 0.050 0.038 0.031	1003 1067 1171 1317 1365 1426 1364 1269	1028 1451 2699 4865 8834 6468 4233 2723
2.5	2 4 6 8 10 12 14 16	3.76 5.95 9.32 21.71 22.59 22.21 21.39 9.03	7.04 9.43 21.71 23.06 24.20 23.70 22.66 21.66	0.024 0.027 0.036 0.045 0.053 0.045 0.035	1018 1074 1180 1320 1418 1411 1346 1252	1035 1380 2494 4504 6728 5616 3734 2349

Add. Conc. (Z)	Ht. above Burner (cm)	Pyrometer 1 flame 2		Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}})$
20	2 4 6 8 10 12 14	2.59 4.96 5.57 21.64 22.12 21.90 21.51 8.81	5.30 9.06 10.24 22.97 23.59 23.31 22.81 21.61	0.021 0.028 0.032 0.044 0.045 0.042 0.038 0.029	969 1052 1048 1315 1396 1380 1341 1232	7766 1338 1515 4342 5416 4873 4006 2308
15	2	2.79	5.58	0.022	972	820
	4	4.92	8.99	0.027	1060	1322
	6	8.28	21.49	0.033	1169	2217
	8	21.54	22.84	0.043	1303	4117
	10	22.044	23.48	0.044	1388	5205
	12	21.88	23.28	0.041	1385	4813
	14	21.44	22.73	0.036	1347	3848
	16	8.87	21.62	0.030	1223	2332
10	2	2.98	5.86	0.023	974	864
	4	6.60	12.87	0.037	1060	1812
	6	7.68	21.35	0.032	1153	2856
	8	21.55	22.86	0.042	1314	4136
	10	22.08	23.53	0.044	1396	5296
	12	21.86	23.26	0.040	1391	4759
	14	21.44	22.73	0.035	1356	3830
	16	13.49	22.28	0.052	1159	3393
5	2 4 6 8 10 12 14 16	2.98 4.81 7.62 21.57 22.15 21.87 21.34 8.58	5.86 8.81 21.33 22.88 23.63 23.26 22.59 21.56	0.022 0.026 0.031 0.042 0.043 0.039 0.033	986 1064 1160 1318 1418 1400 1356 1246	858 1292 2032 4178 5459 4748 3604 2232
2.5	2	3.22	6.21	0.022	1004	907
	4	4.94	9.05	0.026	1072	1323
	6	7.78	21.37	0.031	1168	2073
	8	21.53	22.84	0.041	1319	4088
	10	22.13	23.61	0.043	1415	5418
	12	21.75	23.11	0.038	1387	4478
	14	21.17	22.38	0.031	1343	3281
	16	8.39	21.52	0.026	1251	2177

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromete 1 flame	r Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kca1.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}}$
20	2 4 6 8 10 12 14	2.64 5.22 8.96 21.34 21.73 21.27 8.92 6.32	5.37 9.56 21.64 22.60 23.10 22.51 21.64 12.14	0.021 0.029 0.084 0.035 0.034 0.028 0.021 0.019	973 1057 1186 1336 1425 1400 1346 1252	787 1410 2392 3646 4390 3409 2240 1597
15	2 4 6 8 10 12 14 16	3.06 5.17 9.21 21.42 21.91 21.36 10.10 6.71	5.97 9.45 21.69 22.69 23.32 22.63 21.87 13.49	0.023 0.029 0.034 0.037 0.037 0.030 0.024	980 1054 1195 1332 1428 1396 1343 1241	879 1396 2450 3808 4814 3611 2540 1714
10	2 4 6 8 10 12 14 16	3.06 5.15 8.53 21.46 21.92 21.48 10.65 7.55	5.97 9.41 21.55 22.74 23.33 22.77 21.96 21.33	0.023 0.028 0.033 0.039 0.039 0.033 0.026 0.023	980 1063 1179 1321 1411 1384 1332 1254	879 1384 2276 3911 4867 3866 2676 1942
5	2 4 6 8 10 12 14 16	3.16 5.11 8.05 21.47 22.00 21.64 12.51 7.52	6.13 9.36 21.44 22.75 23.43 22.98 22.22 21.32	0.024 0.027 0.032 0.040 0.041 0.036 0.029 0.023	976 1072 1169 1314 1407 1386 1338	906 1370 2150 3939 5068 4234 3033 1934
2.5	2 4 6 8 10 12 14 16	3.20 5.04 7.62 21.32 22.01 21.56 12.37 7.82	6.20 9.22 21.33 22.54 23.45 22.87 22.20 21.39	0.022 0.026 0.031 0.060 0.042 0.037 0.030 0.025	1003 1078 1160 1152 1401 1360 1324 1239	905 1345 2032 3846 5123 4081 3027 2028

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometer 1 flame 2		Emiss.	C)	Rad. (Kcal. n-2h1 ster1)
20	2 4 6 8 10 12 14 16	2.82 5.22 8.27 21.70 22.31 22.15 11.76 10.56	5.63 9.56 21.49 23.05 23.82 23.62 22.11 21.94	0.022 0.029 0.035 0.047 0.050 0.047 0.045 0.035	974 1057 1150 1304 1390 1384 1176	826 1410 2228 4509 5937 5506 3084 2761
15	2 4 6 8 10 12 14 16	2.74 4.92 7.90 21.64 22.28 22.03 21.55 10.53	5.50 8.99 21.40 22.96 23.80 23.47 22.85 21.94	0.022 0.028 0.033 0.045 0.048 0.044 0.039	968 1049 1153 1306 1401 1388 1339 1247	809 1329 2120 4343 5855 5195 4087 2734
10	2 4 6 8 10 12 14 16	3.00 4.93 7.43 21.51 22.17 21.97 21.45	5.89 9.01 21.29 22.80 23.65 23.39 22.74 21.98	0.023 0.027 0.032 0.043 0.046 0.042 0.036	976 1060 1142 1297 1396 1392 1348 1274	868 1325 1992 4058 5547 5016 3863 2757
5	2 4 6 8 10 12 14 16	3.09 4.64 7.48 21.53 22.15 21.88 21.38 9.31	6.02 8.49 21.30 22.83 23.62 23.28 22.64 21.72	0.022 0.026 0.031 0.042 0.044 0.040 0.034	994 1053 1154 1309 1408 1394 1353	1996 4087 5460 4796 3692
2.5	2 4 6 8 10 12 14 16	3.31 4.79 7.78 21.51 22.15 21.88 21.24 8.30	6.36 8.76 21.37 22.81 23.63 23.28 22.47 21.50	0.022 0.026 0.031 0.041 0.043 0.039 0.032		1285 2073 4046 5449 4775

Add. Conc. (%)	Ht. above Burner (cm)	Pyromete 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}})$
20	2 4 6 8 10 12 14	2.61 5.05 21.23 22.20 22.99 22.68 22.90 21.90	5.32 9.22 22.43 23.66 24.68 24.28 24.57 23.28	0.022 0.029 0.056 0.068 0.079 0.070 0.074	958 1047 1158 1265 1336 1332 1346 1272	785 1366 3646 5911 8229 7205 7894 5048
15	2 4 6 8 10 12 14	3.09 5.24 1296 22.01 23.08 23.22 22.55 22.00	6.03 9.59 22.24 23.42 24.81 25.00 24.13 23.41	0.024 0.030 0.051 0.060 0.076 0.074 0.063 0.058	971 1048 1158 1275 1364 1394 1348 1284	892 1420 3321 5347 8465 8872 6760 5298
10	2 4 6 8 10 12 14	3.38 5.35 10.34 21.74 22.95 22.77 22.17 21.37	6.46 9.81 21.90 23.09 24.65 24.42 23.65 22.63	0.025 0.030 0.042 0.051 0.068 0.060 0.050 0.042	980 1055 1164 1283 1384 1401 1367 1279	958 1448 2778 4639 7967 7319 5613 3782
5	2 4 6 8 10 12 14	3.38 5.42 9.18 21.65 22.73 22.36 21.69 10.44	6.46 9.94 21.68 22.98 24.37 23.90 23.04 21.93	0.024 0.029 0.037 0.047 0.059 0.048 0.040	992 1068 1167 1294 1401 1415 1358 1265	953 1458 2469 4400 7197 6048 4395 2695
2.5	2 4 6 8 10 12 14	3.34 5.35 8.54 21.58 22.60 22.10 21.39 9.05	6.40 9.81 21.55 22.89 24.20 23.56 22.65 21.66	0.023 0.028 0.034 0.044 0.054 0.044 0.035 0.028	1000 1075 1170 1303 1412 1400 1345 1252	

Add. Conc. (%)	Ht. above Burner (cm)		er Output 2 flames	Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{-2}{\text{h}}} \cdot -1$ $\frac{\text{ster} \cdot -1}{}$
20	2 4 6 8 10 12 14	2.83 5.52 21.32 22.68 23.77 24.32 23.92 22.63	5.64 10.14 22.54 24.27 25.69 26.43 25.89 24.18	0.022 0.031 0.058 0.086 0.113 0.130 0.115 0.095	975 1055 1164 1262 1320 1338 1332 1220	828 1496 3835 7413 11287 13595 11860 7337
15	2 4 6 8 10 12 14	3.07 5.69 11.76 22.61 23.79 24.16 23.45 22.29	5.99 10.51 22.11 24.18 25.74 26.23 25.29 23.76	0.023 0.031 0.045 0.080 0.106 0.117 0.096 0.080	980 1064 1176 1276 1345 1356 1335	882 1540 3084 7144 11285 12804 9960 6258
10	2 4 6 8 10 12 14	3.38 6.12 12.97 22.60 23.70 23.62 22.91 21.83	6.46 11.51 22.24 24.17 25.63 25.52 24.59 23.19	0.024 0.033 0.050 0.076 0.095 0.093 0.075	992 1069 1164 1291 1372 1368 1344 1239	953 1663 3314 77057 10790 10481 7953 4948
5	2 4 6 8 10 12 14	3.69 5.78 11.15 22.31 23.21 22.69 22.13 21.20	6.94 10.70 22.03 23.81 25.00 24.31 23.59 22.40	0.025 0.031 0.044 0.063 0.071 0.058 0.050 0.041	1002 1069 1170 1308 1408 1400 1359 1252	1026 1562 2962 6118 8811 7061 5504 3447
2.5	2 6 8 10 12 14	3.52 5.22 10.19 21.85 22.86 22.49 21.68 10.82	6.69 9.56 21.87 23.23 24.55 24.06 23.02 21.99	0.023 0.027 0.039 0.048 0.059 0.052 0.041 0.032	1013 1078 1183 1324 1423 1407 1347 1265	2720 4843

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometer 1 flame 2		Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}}$
20	2 4 6 8 10 12 14	2.45 5.06 10.25 22.08 23.01 22.78 22.29 21.42	5.09 9.25 21.88 23.51 24.73 24.73 23.78 22.68	0.020 0.029 0.042 0.058 0.068 0.065 0.061	971 1048 1161 1298 1393 1374 1316 1218	743 1370 2759 5485 8137 7426 6044 3990
15	2 4 6 8 10 12 14 16	2.85 5.45 9.55 21.93 23.31 22.46 21.94 21.25	5.67 10.00 21.75 23.33 25.13 24.02 23.34 22.47	0.022 0.030 0.039 0.053 0.072 0.056 0.051	976 1060 1163 1303 1418 1376 1318 1240	832 1473 2574 5074 9140 6423 5074 3576
10	2 4 6 8 10 12 14	3.18 5.22 8.70 21.79 22.37 22.13 21.62 11.77	6.15 9.56 21.58 23.16 23.90 23.60 22.95 22.12	0.023 0.029 0.036 0.048 0.052 0.047 0.042	988 1057 1158 1313 1387 1382 1328 1249	904 4410 2344 4718 6128 5475 4280 3576
5	2 4 6 8 10 12 14 16	3.22 4.98 8.36 21.65 22.24 22.00 21.35 9.15	6.22 9.11 21.51 22.98 23.74 23.43 22.61 21.68	0.023 0.027 0.033 0.044 0.047 0.043 0.035 0.029	992 1064 1172 1316 1401 1389 1338 1244	913 1338 2236 4360 5733 5096 3660 2387
2.5	2 4 6 8 10 12 14 16	3.09 4.93 8.28 21.59 22.68 21.87 21.27	6.02 9.02 21.49 22.90 24.31 23.27 22.51 22.11	0.022 0.026 0.032 0.042 0.057 0.040 0.033 0.045	994 1072 1179 1320 1406 1392 1343 1176	880 1320 2207 4204 7033 4768 3492 3084

Add. Conc. (%)	Ht.  above Burner (cm)	Pyrometer 1 flame 2	Output flames	Emiss.	Temp. (K	d. 2 <sub>h</sub> -1 er -1
20	2 4 6 8 10 12 14 16	2.50 5.17 10.14 22.09 22.97 22.76 22.28 21.33	5.16 9.45 21.86 23.53 24.68 24.41 23.77 22.56	0.020 0.029 0.042 0.058 0.069 0.066 0.061	975 1054 1158 1300 1383 1366 1315 1212	753 1396 2735 5519 8053 7395 6020 3779
15	2 4 6 8 10 12 14 16	2.87 5.41 9.64 21.95 22.69 22.43 21.99 21.16	5.70 9.92 21.77 23.36 24.32 23.97 23.41 22.36	0.022 0.030 0.040 0.053 0.060 0.057 0.052 0.043	978 1052 1158 1308 1389 1364 1321 1229	837 1462 2604 5136 7111 6349 5215 3400
10	2 4 6 8 10 12 14 16	3.15 5.22 9.11 21.72 22.35 22.13 21.56 11.10	6.12 9.56 21.67 23.07 23.88 23.59 22.87 22.03	0.023 0.029 0.037 0.048 0.052 0.048 0.042	987 1057 1164 1300 1384 1373 1316 1243	900 1410 2452 4558 6080 5473 4153 2869
5	2 4 6 8 10 12 14 16	3.29 4.94 8.54 21.55 22.19 22.47 21.35 10.18	6.32 9.04 21.55 22.85 23.68 24.02 22.61 21.88	0.023 0.027 0.034 0.044 0.047 0.056 0.036	996 1061 1170 1297 1392 1376 1328 1257	927 1329 2289 4153 5614 6435 3676 2639
2.5	2 4 6 8 10 12 14	3.15 4.81 7.96 21.52 22.17 21.85 12.55 8.90	6.12 8.81 21.42 22.81 23.65 23.24 22.22 21.63	0.022 0.026 0.032 0.042 0.045 0.040 0.033 0.028	999 1064 1165 1307 1404 1388 1294 1247	894 1292 2126 4062 5531 4723 3089 2319

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometer 1 flame 2		Emiss.	Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{m^{-2}h}{-1}}$
20	2 4 6 8 10 12 14	2.60 5.04 8.98 21.83 22.62 22.35 21.90 10.96	5.31 9.21 21.64 23.20 24.22 23.87 23.30 22.01	0.020 0.028 0.038 0.053 0.061 0.056 0.050	983 1056 1152 1286 1372 1357 1319 1230	772 1358 2430 4861 6928 6141 4984 2852
15	2 4 6 8 10 12 14	2.79 5.31 8.41 21.73 22.42 22.08 21.60 10.41	5.58 9.72 21.52 23.08 23.97 23.53 22.91 21.91	0.021 0.030 0.036 0.050 0.055 0.048 0.043	984 1052 1147 1288 1376 1364 1314 1215	815 1438 2271 4605 6308 5357 4235 2739
10	2 4 6 8 10 12 14 16	3.26 5.15 7.88 21.62 22.27 21.86 21.32 9.06	6.28 9.41 21.40 22.94 23.78 23.25 22.56 21.66	0.023 0.028 0.033 0.046 0.050 0.042 0.036 0.030	995 1063 1152 1296 1384 1372 1321 1230	923 1384 2115 4324 5858 4770 3610 2377
5	2 4 6 8 10 12 14 16	3.39 5.15 8.07 21.57 22.16 21.74 21.27 8.49	6.47 9.43 21.44 22.88 23.64 23.10 22.50 21.54	0.023 0.027 0.033 0.044 0.046 0.040 0.034 0.029	1004 1074 1160 1302 1395 1368 1331 1219	949 1380 2163 4204 5526 4499 3493 2230
2.5	2 4 6 8 10 12 14 16	3.31 4.85 8.05 11.49 22.13 21.76 21.25 7.77	6.36 8.88 21.44 22.09 23.60 21.13 22.48 21.38	0.022 0.026 0.032 0.027 0.044 0.039 0.033	1011 1067 1169 1341 1406 1380 1338	928 1301 2150 2846 5429 4525 3451 2025

Add. Conc. (%)	Ht. above Burner (cm)	Pyromete 1 flame	er Output 2 flames	Emiss.	Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14 16	2.49 4.75 21.16 22.34 23.32 24.02 23.60 22.66	5.15 8.68 22.35 23.83 25.11 26.05 25.49 24.24	0.021 0.028 0.055 0.075 0.093 0.105 0.096 0.081	961 1039 1152 1254 1327 1377 1355 1278	757 1288 3517 6331 9458 12089 10465 7278
15	2 4 6 8 10 12 14	2.75 4.87 11.73 22.19 23.26 23.62 21.18 22.39	5.52 8.90 22.10 23.65 25.05 25.52 24.95 23.90	0.022 0.029 0.050 0.068 0.085 0.094 0.083 0.074	968 1036 1143 1264 1351 1364 1348 1267	811 1324 3119 5886 9175 10491 8889 6459
10	2 4 6 8 10 12 14 16	3.11 5.32 10.72 22.05 22.42 23.29 22.81 22.04	6.05 9.75 21.96 23.47 23.95 25.10 24.47 23.46	0.023 0.030 0.045 0.061 0.063 0.077 0.068 0.061	984 1053 1152 1276 1327 1391 1363 1274	891 1441 2884 5447 6407 9161 7560 5425
5	2 4 6 8 10 12 14 16	3.53 5.17 9.33 21.76 22.86 22.69 22.10 21.27	6.69 9.46 21.71 23.12 24.54 24.31 23.55 22.50	0.024 0.028 0.038 0.051 0.064 0.058 0.050	1002 1064 1164 1287 1392 1400 1354 1259	985 1391 2513 4687 7644 7061 5440 3590
2.5	2 4 6 8 10 12 14 16	3.45 5.05 8.60 21.59 22.64 22.29 21.67 9.61	6.52 9.23 21.56 22.90 24.26 23.80 23.01 21.77	0.023 0.027 0.034 0.045 0.056 0.049 0.042	1006 1068 1172 1297 1406 1394 1337 1239	956 1354 2304 4247 6910 5875 4384 2514

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometer 1 flame 2	Output flames	Emiss.	Flame Temp.	Rad. (Kcal. m <sup>-2</sup> h1 ster1)
20	2 4 6 8 10 12 14 16	22.74 24.05 24.62 23.81	4.79 8.23 22.51 24.35 26.09 26.89 25.75 24.36	0.020 0.025 0.058 0.085 0.109 0.121 0.108 0.088	954 1055 1159 1275 1368 1400 1340 1264	704 1207 3785 7574 12260 14703 11362 7633
15	2 4 6 8 10 12 14 16	22.42 23.87 24.05 23.38	5.25 8.55 22.55 23.93 25.86 26.11 25.20 24.11	0.021 0.027 0.060 0.073 0.100 0.102 0.090 0.078	967 1044 1154 1276 1376 1392 1348 1275	770 1263 3863 6519 11491 12182 9639 6951
10	2 4 6 8 10 12 14	22.25 23.27 23.17 22.38	5.67 9.16 21.53 23.73 25.07 24.94 23.91 22.80	0.022 0.028 0.033 0.066 0.078 0.075 0.060 0.049	976 1055 1176 1283 1383 1383 1338 1254	832 1351 2261 6004 9103 8753 6275 4136
5	2 4 6 8 10 12 14	21.76 22.67 22.18 21.46	7.17 9.44 21.82 23.12 24.29 23.65 22.74 21.89	0.026 0.028 0.040 0.050 0.059 0.050 0.030	1001 1064 1165 1293 1391 1368 1331 1269	1064 1388 2657 4671 7020 5624 3904 2635
2.5	2 4 8 10 12 14	21.60 22.24 21.81 21.23	6.70 9.62 21.56 22.92 23.74 23.20 22.45 21.50	0.025 0.028 0.034 0.044 0.047 0.040 0.034 0.027	992 1069 1172 1307 1400 1381 1323 1234	993 1411 2299 4255 5722 4650 3423 2162

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame		t Emîss.	Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{m^{-2}h}{\text{.}}^{-1}}$
20	2 4 6 8 10 12 14	2.41 5.02 9.31 21.86 22.51 22.29 21.85 11.79	5.03 9.18 21.71 23.24 24.09 23.80 23.23 22.12	0.020 0.028 0.039 0.053 0.057 0.054 0.050	968 1056 1155 1292 1378 1360 1308 1207	736 1355 2517 4931 6575 5956 4855 3053
15	2 4 6 8 10 12 14	2.85 5.09 8.91 21.62 23.01 22.17 21.67 10.90	5.67 9.30 21.63 22.93 24.72 23.64 23.01 22.00	0.022 0.029 0.037 0.048 0.071 0.049 0.044	976 1049 1157 1280 1377 1373 1320 1228	832 1376 2404 4840 8175 5587 4404 2840
10	2 4 6 8 10 12 14	3.14 5.02 8.23 21.61 22.34 22.11 21.59 10.92	6.10 9.18 21.48 22.93 23.87 23.57 22.91 22.00	0.023 0.028 0.035 0.046 0.050 0.046 0.040	986 1056 1148 1294 1396 1386 1339 1248	897 1355 2218 4306 6029 5410 4192 2822
5	2 4 6 8 10 12 14 16	3.32 5.01 8.03 21.58 22.25 22.05 21.53 9.43	6.37 9.16 21.43 22.89 23.75 23.49 22.83 21.74	0.023 0.027 0.033 0.043 0.047 0.043 0.036 0.029	999 1065 1159 1304 1402 1399 1364 1255	934 1345 2153 4221 5744 5215 1010 2453
2.5	2 4 6 8 10 12 14 16	3.18 4.94 7.84 21.63 22.23 21.91 21.29 7.23	6.16 9.05 21.39 22.96 23.73 23.33 22.54 21.24	0.022 0.026 0.032 0.043 0.045 0.040 0.033 0.023	1001 1072 1160 1321 1415 1400 1347 1238	901 1323 2098 4312 5670 4870 3527 1862

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	THE RESERVE THE PERSON NAMED IN COLUMN TWO IS NOT THE PERSON NAMED IN COLUMN TWO IS NAMED IN COLUMN TWIND TWO IS NAMED IN COLUMN TWO IS NAMED IN COLUMN TWO IS NAMED IN		Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{m}}}$
	(0.11)					ster. 1)
ر 5	2	3.24	6.25	0.022	1005	912
	4	4.94	9.05	0.026	1072	1323
	4 6	7.85	21.39	0.031	1171	2092
	8	21.35	22.60	0.040	1291	3714
	10	22.05	23.50	0.044	1391	5235
	12	21.79	23.16	0.039	1385	4578
	14	11.62	22.11	0.030	1308	2907
	16	7.31	21.26	0.025	1215	1902
215	2	3.24	6.25	0.022	1005	912
		4.67	8.56	0.026	1056	1258
	4 6 8	8.00	21.43	0.031	1177	2129
	8	21.41	22.68	0.041	1295	3846
	10	22.05	23.50	0.043	1400	5225
	12	21.67	23.01	0.038	1372	4316
	14	12.43	22.21	0.031	1313	3047
	16	7.72	21.36	0.027	1209	2023

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame		Flame Temp. (°C)	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.30 4.35 21.27 22.63 23.93 24.55 23.92 22.79	4.87 0.020 8.00 0.025 22.48 0.058 24.21 0.082 25.92 0.112 26.79 0.124 25.90 0.112 24.41 0.086	959 1046 1154 1270 1344 1383 1342 1278	715 1175 3734 7217 11877 14472 11830 7727
15	2 4 6 8 10 12 14 16	2.59 4.56 21.29 22.34 23.77 24.03 23.43 22.59	5.30 0.021 8.36 0.026 22.51 0.058 23.84 0.073 25.72 0.100 26.08 0.102 25.27 0.091 24.15 0.077	969 1048 1159 1264 1363 1389 1351 1284	776 1231 3785 6319 11117 12089 9823 7033
10	2 4 6 8 10 12 14	2.89 4.89 21.23 22.19 23.26 24.14 22.58 21.73	5.73 0.022 8.94 0.028 22.43 0.054 23.66 0.065 25.06 0.079 26.26 0.093 24.17 0.065 23.07 0.053	980 1048 1169 1279 1377 1438 1343 1268	841 1322 3627 5852 9096 12375 6879 4636
5	2 4 6 8 10 12 14	3.46 5.45 11.04 13.06 22.67 22.23 21.57 10.42	6.58 0.024 10.00 0.030 22.01 0.045 22.25 0.047 24.29 0.059 23.73 0.048 22.88 0.040 21.92 0.032	997 1060 1160 1185 1391 1392 1334 1254	970 1473 2950 3300 7020 5722 4142 2701
2.5	2 4 6 8 10 12 14	3.72 5.61 9.40 21.71 22.37 21.96 21.32 8.15	6.97 0.025 10.33 0.030 21.73 0.038 23.05 0.048 23.91 0.050 23.38 0.043 22.57 0.035 21.47 0.026	1004 1069 1166 1297 1402 1383 1332 1240	1031 1512 2530 4530 6111 5018 3610 2118

Run No. L18 Additive water + 0.005M surfactant

Add. Conc. (%)	Ht. above Burner (cm)		er Output 2 flames		Flame Temp.	Rad. (Kcal. M <sup>-2</sup> h. <sup>-1</sup> ster. <sup>-1</sup> )
20	2 4 6 8 10 12 14 16	2.46 4.21 6.90 21.20 21.86 21.50 11.43 6.04	7.77 21.15 22.42 23.25 22.80 22.08	0.020 0.024 0.029 0.038 0.042 0.036 0.029	972 1048 1148 1280 1372 1359 1315 1204	745 1136 1834 3428 4770 3963 2862 1550
15	2 4 6 8 10 12 14 16	2.71 4.51 8.38 21.26 21.89 21.56 11.32 6.46	8.27 21.51 22.49 23.30 22.87 22.07	0.021 0.025 0.034 0.039 0.042 0.037 0.030 0.023	978 1056 1164 1282 1379 1360 1300 1198	799 1212 2249 3540 4854 4081 2855 1672
10	2 4 6 8 10 12 14 16	2.83 4.51 6.89 21.37 21.96 21.58 11.94 6.61	5.64 8.27 21.15 22.63 23.38 22.90 22.15 13.06	0.021 0.025 0.030 0.040 0.042 0.037 0.030 0.023	988 1056 1136 1296 1391 1364 1315 1206	824 1212 1838 3760 4997 4121 2960 1709
5	2 4 6 8 10 12 14	3.24 4.65 7.31 21.41 22.43 21.67 13.60 7.31	6.25 8.51 21.26 22.68 23.99 23.02 22.31 21.26	0.022 0.025 0.030 0.040 0.052 0.038 0.031	1005 1065 1156 1304 1398 1372 1332 1215	912 1245 1945 3837 6294 4324 3191 1902
2.5	2 4 6 8 10 12 14		6.16 8.79 21.79 22.75 23.55 23.02 22.36 21.40	0.022 0.026 0.030 0.040 0.043 0.038 0.031	1001 1064 1162 1314 1407 1372 1339 1228	901 1289 1975 3939 5316 4324 3248 2051

Run No. L19 Additive water + 0.01M surfactant

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometo 1 flame	er Output 2 flames	Emiss.	Flame Temp. (°C)	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.50 4.42 6.79 21.24 21.83 21.49 10.88 5.69	8.13 0 13.74 0 22.46 0 23.21 0 22.79 0 22.00 0	.020 .025 .029 .038 .041 .036 .029	975 1051 1142 1287 1375 1356 1300 1199	753 1192 1805 3492 4693 3940 2760 1457
15	2 4 6 8 10 12 14	2.70 4.32 7.12 21.30 21.89 21.61 11.52 4.53	7.96 0 21.22 0 22.53 0 23.30 0 22.94 0 22.09 0	.021 .025 .030 .039 .042 .037 .030	977 1044 1150 1259 1379 1370 1305 1058	797 1169 1910 3607 4854 4186 2890 1218
10	2 4 6 8 10 12 14 16	2.78 4.46 7.24 21.33 21.93 21.59 11.87 6.07	8.19 0 21.24 0 22.57 0 23.34 0 22.92 0 22.14 0	.021 .025 .030 .039 .042 .037 .030	984 1053 1153 1296 1385 1367 1313 1205	813 1201 1927 3666 4930 4154 2949 1557
5	2 4 6 8 10 12 14	3.12 4.74 7.46 21.40 22.02 21.64 12.74 6.49	8.67 0 21.30 0 22.66 0 23.46 0 22.97 0 22.24 0	.022 .026 .030 .040 .043 .037	996 1060 1164 1300 1394 1376 1319	887 1273 1984 3806 5156 4244 3090 1670
2.5	2 4 6 8 10 12 14	3.22 4.85 7.89 21.43 21.97 21.72 13.14 7.27	8.88 0 21.40 0 22.70 0 23.39 0 23.08 0 22.28 0	.022 .026 .031 .040 .043 .038 .031	1004 1067 1172 1307 1384 1382 1325 1226	907 1301 2101 3868 5038 4426 3140 1882

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometo 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{m^{-2}h}{-1}}$ ster1
20	2 4 6 8 10 12 14 16	2.19 4.38 13.61 22.51 23.86 24.77 22.92 23.04	8.05 22.29 24.05 25.81 27.06 24.61	0.020 0.027 0.055 0.081 0.116 0.140 0.074 0.102	950 1026 1143 1256 1322 1264 1350 1256	695 1194 3431 6867 11657 15594 7972 8665
15	2 4 6 8 10 12 14	2.68 4.82 13.13 22.47 23.65 24.23 23.76 22.89	8.81 22.25 24.00 25.54 26.33 25.69	0.022 0.029 0.053 0.076 0.103 0.120 0.120	964 1033 1148 1271 1336 1356 1357 1277	799 1311 3359 6703 10729 13133 11168 8070
10	2 4 6 8 10 12 14	3.07 5.05 12.14 22.43 23.50 23.83 23.18 22.41	9.21 22.15 23.96 25.36 25.81 24.95	0.023 0.031 0.049 0.071 0.090 0.098 0.079	980 1028 1157 1288 1364 1378 1365 1270	882 1377 3183 6539 10025 11305 8834 6513
5	2 4 6 8 10 12 14	3.31 5.31 7.19 22.07 23.07 23.02 22.48 21.51	9.72 21.22 23.51 24.81 24.74 24.05	0.025 0.030 0.040 0.058 0.073 0.069 0.058	976 1052 1064 1297 1377 1389 1367	946 1438 1987 5474 8405 8178 6511 4102
2.5	2 4 6 8 10 12 14 16	3.50 5.15 9.34 21.81 22.68 22.35 21.67 10.18	9.41 21.71 23.18 24.31 23.87 23.01	0.024 0.028 0.037 0.050 0.059 0.051 0.042	1000 1063 1172 1302 1393 1390 1337 1247	977 1384 2508 4777 7060 6056 4384 2651

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometer 1 flame 2		Emiss.	Flame Temp.	Rad. (Kcal. -2h1 ster1)
20	2 4 6 8 10 12 14	2.54 5.42 10.49 22.07 23.22 23.00 22.39 21.36	5.22 9.95 21.92 23.50 25.00 24.71 23.91 22.61	0.020 0.030 0.043 0.059 0.075 0.072 0.064 0.052	978 1059 1160 1291 1389 1372 1316 1207	761 1466 2819 5478 8889 8178 6341 3872
15	2 4 6 8 10 12 14	3.00 5.42 10.13 21.94 22.89 22.72 21.97 21.15	5.89 9.95 21.86 23.34 24.57 24.36 23.38 22.34	0.022 0.030 0.041 0.054 0.067 0.062 0.051 0.044	988 1059 1165 1299 1380 1382 1324 1220	863 1466 2724 5117 7774 7222 5145 3391
10	2 4 6 8 10 12 14	3.30 5.31 8.91 21.84 22.59 22.23 21.64 10.96	6.33 9.72 21.63 23.22 24.19 23.72 22.97 22.01	0.023 0.030 0.037 0.049 0.057 0.050 0.042 0.036	997 1052 1157 1315 1391 1376 1331 1230	930 1438 2404 4886 6782 5746 4314 2852
5	2 4 6 8 10 12 14 16	3.23 5.33 8.54 21.65 22.39 21.99 21.37 10.13	6.23 9.77 21.55 22.98 23.94 23.42 22.63 21.87	0.023 0.029 0.024 0.044 0.051 0.043 0.035	992 1064 1170 1316 1398 1388 1342 1256	916 1437 2289 4360 6174 5087 3697 2628
2.5	2 4 6 8 10 12 14	3.18 4.48 7.90 21.47 22.24 21.82 21.26 8.70	6.16 8.23 21.40 22.75 23.74 23.21 22.50 21.59	0.022 0.025 0.032 0.042 0.046 0.040 0.033 0.028	1001 1055 1163 1297 1409 1383 1340	901 1207 2112 3964 5719 4668 3472 2271

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame		Emîss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{m^{-2}h.}{\text{ster.}^{-1}}})$
20	2 4 6 8 10 12 14 16	2.48 4.84 8.75 21.55 21.94 21.55 11.40 7.20	8.85 0 21.59 0 22.86 0 23.36 0 22.86 0 22.08 0	.020 .027 .034 .040 .039 .034 .026	973 1055 1178 1331 1415 1388 1352 1267	749 1303 2340 4109 4914 4022 2818 1833
15	2 4 6 8 10 12 14	2.74 5.15 8.79 21.56 22.02 21.57 13.04 7.25	9.41 0 21.60 0 22.87 0 23.46 0 22.89 0 22.27 0	0.021 0.028 0.034 0.041 0.041 0.035 0.028	980 1063 1180 1324 1412 1383 1360 1254	805 1384 2350 4145 5127 4085 3088 1857
10	2 4 6 8 10 12 14 16	3.18 5.34 8.58 21.54 22.05 21.69 13.77 7.12	9.78 0 21.56 0 22.85 0 23.50 0 23.04 0 22.33 0	0.022 0.028 0.034 0.042 0.042 0.036 0.029	1001 1074 1172 1312 1408 1395 1357 1263	901 1432 2299 4120 5202 4325 3180 1814
5	2 4 6 8 10 12 14 16	3.25 5.30 8.19 21.47 22.11 21.77 13.70 7.47	9.72 0 21.47 0 22.76 0 23.57 0 23.14 0 22.32 0	0.023 0.027 0.033 0.041 0.042 0.037 0.030 0.023	994 1083 1165 1307 1418 1401 1344 1250	920 1417 2192 3965 5332 4513 3188 1921
2.5	2 4 6 8 10 12 14 16		8.97 0 21.38 0 22.70 0 23.55 0 23.17 0 22.31 0	0.022 0.026 0.032 0.041 0.043 0.038 0.031	1004 1070 1158 1299 1407 1396 1331 1244	907 1313 2084 3886 5316 4582 3184 1893

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometo 1 flame	er Output Emiss. 2 flames	$\frac{\frac{\text{Flame}}{\text{Temp.}}}{\frac{\text{(}^{\circ}\text{C})}{\text{(}^{\circ}\text{C})}} \frac{\frac{\text{Rad.}}{\text{(}^{\circ}\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}})$
20	2 4 6 8 10 12 14	2.57 4.97 8.91 21.55 22.34 22.07 21.50 9.16	5.27 0.021 9.09 0.028 21.63 0.035 22.86 0.038 23.88 0.047 23.53 0.043 22.80 0.038 21.68 0.031	968 772 1052 1342 1175 2388 1349 4086 1420 5989 1404 5275 1339 3982 1223 2410
15	2 4 6 8 10 12 14	3.03 4.66 8.48 21.69 22.25 21.99 21.43 7.24	5.94 0.023 8.52 0.027 21.54 0.033 23.03 0.044 23.76 0.046 23.42 0.042 22.71 0.036 21.25 0.022	978 875 1044 1260 1177 2266 1324 4439 1411 5741 1396 5065 1344 3818 1253 1853
10	2 4 6 8 10 12 14 16	3.10 4.86 8.07 21.61 22.17 21.94 21.26 9.17	6.04 0.023 8.89 0.027 21.44 0.032 22.93 0.043 23.66 0.045 23.35 0.040 22.49 0.034 21.69 0.029	983 888 1056 1309 1170 2154 1316 4261 1405 5542 1404 4917 1329 3479 1245 2392
5	2 4 6 8 10 12 14	3.24 4.76 7.60 21.53 22.16 21.83 21.22 8.44	6.25 0.022 8.72 0.026 21.33 0.031 22.83 0.042 23.63 0.044 23.22 0.039 22.45 0.032 21.53 0.028	1005 912 1061 1279 1160 2027 1309 4087 1410 5481 1394 4676 1343 3387 1228 2209
2.5	2 4 6 8 10 12 14 16	3.16 4.88 7.60 21.51 22.29 21.73 21.14 7.75	6.13 0.022 8.93 0.026 21.33 0.031 22.81 0.041 23.80 0.053 23.09 0.038 22.35 0.031 21.37 0.026	1000 896 1068 1307 1160 2027 1315 4046 1367 5950 1384 4452 1337 3236 1223 2021

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame			Flame Temp. (°C)	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{m^{-2}h}{-1}}$ ster1
20	2 4 6 8 10 12 14 16	2.22 4.74 7.26 21.47 21.71 21.48 10.45 6.71	4.77 8.67 21.25 22.76 23.06 22.78 21.93 13.49	0.019 0.026 0.027 0.037 0.034 0.031 0.024 0.020	967 1060 1188 1343 1420 1408 1354 1257	697 1273 1908 3916 4341 3840 2611 1703
15	2 4 6 8 10 12 14	2.32 4.84 7.89 21.51 21.83 21.54 10.30 7.43	4.91 8.85 21.40 22.81 23.22 22.85 21.91 21.30	0.019 0.027 0.030 0.038 0.036 0.032 0.024	975 1055 1183 1340 1423 1408 1349 1263	715 1303 2092 3998 4622 3963 2580 1900
10	2 4 6 8 10 12 14 16	2.67 4.94 7.91 21.49 21.93 21.63 11.03 7.19	5.41 9.04 21.41 22.79 23.35 22.97 22.03 21.23	0.020 0.027 0.031 0.038 0.038 0.034 0.025	988 1061 1173 1337 1423 1404 1356 1251	786 1329 2106 3966 4879 4179 2736 1842
5	2 4 6 8 10 12 14	3.00 5.06 8.00 21.53 21.97 21.70 11.52 7.65	5.89 9.26 21.43 22.83 23.39 23.05 22.10 21.35	0.021 0.027 0.031 0.040 0.040 0.036 0.027 0.024	1000 1068 1177 1327 1410 1397 1342 1244	857 1357 2129 4068 4983 4350 2852 1976
2.5	2 4 6 8 10 12 14 16	3.19 4.94 8.30 21.55 21.99 21.70 13.70 7.91	6.18 9.05 21.50 21.86 23.43 23.06 22.32 21.41	0.022 0.026 0.032 0.041 0.041 0.038 0.030	1002 1072 1180 1322 1406 1388 1344 1230	903 1323 2212 4120 5059 4377 3188 2060

Run No. L25 Additive trichloro-trifluoroethane

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames		Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kca1.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}})$
20	2 4 6 8 10 12 14	2.73 4.45 7.62 21.45 22.00 21.75 12.42 7.63	5.49 8.17 21.33 ·22.72 23.44 23.11 22.21 21.34	0.021 0.025 0.031 0.039 0.040 0.037 0.029 0.024	980 1052 1160 1319 1417 1396 1336 1244	803 1198 2032 3888 5068 4462 3021 1971
15	2 4 6 8 10 12 14 16	2.80 4.79 7.76 21.52 22.00 21.69 21.13 6.91	5.60 8.76 21.37 22.82 23.44 23.04 22.34 21.16	0.021 0.026 0.030 0.040 0.041 0.037 0.030 0.023	985 1063 1177 1325 1408 1386 1347 1222	817 1285 2060 4052 5078 4352 3207 1784
10	2 4 6 8 10 12 14	3.12 4.65 7.43 21.46 21.70 21.77 21.16 8.07	6.07 8.51 21.29 22.74 23.04 23.14 22.37 21.45	0.022 0.026 0.030 0.040 0.046 0.038 0.031	996 1054 1162 1312 1311 1391 1341 1237	887 1252 1975 3924 4494 4521 3268 2100
5	2 4 6 8 10 12 14	3.31 4.81 7.78 21.45 22.00 21.73 13.42 7.86	6.40 8.81 21.37 22.73 23.44 23.09 22.30 21.40	0.022 0.026 0.031 0.040 0.043 0.038 0.031 0.024	1012 1064 1168 1311 1391 1384 1329 1254	933 1292 2073 3908 5116 4443 3172 2026
2.5	2 4 6 8 10 12 14 16	3.19 4.93 7.78 21.47 22.07 21.70 12.93 8.09	21.37 22.76 23.52 23.06	0.022 0.026 0.031 0.041 0.043 0.038 0.031 0.026	1002 1072 1168 1307 1402 1379 1322 1238	903 1320 2073 3965 5255 4392 3115 2104

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\underset{m}{\underline{\text{Rad}}}.}{\underset{m}{\underline{\text{Kcal}}}.}$
						ster. 1)
20	2	2.59		0.020	982	770
	4	4.30		0.024	1055	1158
	6 8	66.77		0.029	1141	1801
	10	21.34 21.81		0.040	1289	3699
	12	21.48		0.042 0.037	1364 1344	4678 3931
	14	11.52		0.037	1305	2890
	16	5.28		0.029	1060	1423
15	2	2.55		0.020	979	763
	4	4.56		0.025	1060	1224
	6	7.22		0.030	1152	1923
	8	21.40		0.041	1293	3830
	10 12	21.86 21.54		0.042	1373	4789
	14	11.98		0.037 0.030	1356 1316	4049 2967
	16	6.43		0.022	1211	1656
10	2	2.88		0.021	992	834
	4	4.37		0.025	1048	1181
	6	7.34		0.030	1158	1953
	8	21.40		0.041	1293	3830
	10	21.95		0.043	1381	4999
	12	21.67 12.83		0.038	1372	4324
	14 16	7.33		0.031 0.024	1320 1229	3103 1898
5	2	3.02	5.92	0.021	1002	862
	4	4.76		0.026	1061	1279
	6	7.78		0.031	1168	2073
	8	21.42		0.041	1297	3870
	10	22.24		0.047	1400	5722
	12	21.66		0.038	1369	4291
	14	21.15		0.031	1338	3242
	16	6.51	12.70	0.022	1215	1674
2.5	2	3.15		0.022	999	894
	4	4.84		0.026	1066	1298
	6	7.98		0.031	1176	2124
	8	21.46		0.041	1305	3949
	10 12	22.02 21.67		0.043	1394	5156
	14	13.36		0.038 0.031	1372	4234
	16	7.19		0.031	1328 1222	3165 1862
	10			01024	1444	1002

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss.	Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.54 4.18 7.19 21.38 21.89 21.54 12.30 6.54	7.73 0 21.23 0 22.63 0 23.29 0 22.85 0	0.020 0.024 0.030 0.040 0.042 0.037 0.031	978 1047 1151 1296 1377 1356 1311 1216	761 1131 1914 3768 4836 4049 3029 1681
15	2 4 6 8 10 12 14 16	2.65 4.35 7.39 21.36 21.93 21.60 11.99 7.52	8.00 0 21.28 0 22.61 0 23.35 0 22.92 0 22.15 0	0.021 0.025 0.030 0.040 0.042 0.038 0.031	974 1046 1160 1292 1386 1358 1304 1238	789 1175 1966 3739 4940 4175 2980 1942
10	2 4 6 8 10 12 14	2.65 4.50 7.06 21.46 22.03 21.66 12.79 6.93	8.25 0 21.20 0 22.74 0 23.47 0 22.99 0 22.24 0	0.021 0.025 0.024 0.041 0.043 0.038 0.031	974 1056 1176 1304 1396 1369 1320 1223	789 1210 1838 3933 5175 4291 3096 1788
5	2 4 6 8 10 12 14	3.05 4.87 7.85 21.43 22.06 21.64 13.48 7.50	8.90 0 21.39 0 22.70 0 23.51 0 22.97 0	0.022 0.026 0.031 0.041 0.043 0.038 0.031	992 1068 1171 1299 1400 1366 1330 1237	874 1304 2092 3886 5235 4258 3178 1938
2.5	2 4 6 8 10 12 14	3.31 4.91 7.78 21.48 22.09 21.67 13.36 8.30	8.97 0 21.37 0 22.77 0 23.55 0 23.02 0	0.022 0.026 0.031 0.041 0.043 0.038 0.031	1011 1070 1168 1309 1407 1372 1328 1247	928 1313 2073 3989 5316 4324 3165 2154

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromet 1 flame	er Output 2 flames		Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{\text{m}^{-2}\text{h}}{\text{ster}}}$
20	2 4 6 8 10 12 14	2.20 4.06 7.28 21.46 22.15 21.79 21.23 7.82	7.53 21.25 22.74 23.63 23.16 23.46	0.019 0.023 0.031 0.043 0.048 0.042 0.035 0.028	965 1051 1145 1288 1378 1359 1313 1202	694 11097 1947 3968 5537 4624 3440 2058
15	2 4 6 8 10 12 14 16	2.26 4.15 7.14 21.52 22.08 21.77 21.20 8.60	7.67 21.21 22.81 23.53 23.14 22.42	0.019 0.024 0.030 0.044 0.045 0.040 0.034	970 1044 1148 1289 1388 1373 1317 1202	704 1123 1901 4094 5313 4561 3376 2268
10	2 4 6 8 10 12 14	2.67 4.59 8.01 21.49 22.08 21.79 13.61 7.90	8.40 21.43 22.78 23.54 23.16 22.31	0.020 0.026 0.034 0.042 0.044 0.039 0.033 0.028	988 1050 1148 1302 1397 1385 1310 1205	786 1237 2155 4013 5316 4578 3217 2075
5	2 4 6 8 10 12 14	2.87 4.71 7.77 21.54 22.10 21.71 12.88 7.85	8.63 21.37 22.84 23.57 23.07 22.25	0.022 0.026 0.032 0.042 0.043 0.037 0.031	978 1058 1157 1311 1409 1390 1321 1215	837 1267 2079 4103 5346 4394 3109 2054
2.5	2 4 6 8 10 12 14	2.87 5.05 7.71 21.52 22.09 21.87 12.77 7.60	9.23 21.36 22.81 23.55 23.27 22.24	0.021 0.027 0.031 0.041 0.043 0.040 0.030 0.025	991 1068 1164 1316 1407 1392 1195 1228	832 1354 2055 4054 5316 4777 3082 1972

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output E 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{-2}{\text{m-1}}}$
20	2 4 6 8 10 12 14	2.53 4.94 21.38 22.87 23.82 24.44 23.73 22.63	9.04 0. 22.62 0. 24.53 0. 25.81 0. 26.67 0. 25.67 0.	021 027 055 079 090 100 092	964 1061 1192 1319 1408 1448 1387 1291	765 1329 3929 7875 11147 13633 10841 7150
15	2 4 6 8 10 12 14	2.91 4.96 21.26 22.65 23.52 24.02 23.21 22.34	9.06 0. 22.47 0. 24.25 0. 25.40 0. 26.10 0. 24.98 0.	023 027 054 070 080 086 079 067	968 1062 1175 1328 1409 1452 1369 1292	848 1332 3684 7133 9946 11812 8921 6247
10	2 4 6 8 10 12 14	3.15 5.10 11.68 22.27 23.17 23.91 22.42 21.71	9.34 0. 22.10 0. 23.76 0. 24.95 0. 25.89 0. 23.96 0.	024 028 045 061 069 110 059 050	975 1060 1175 1313 1413 1347 1350 1283	904 1374 3070 5996 8661 11757 6356 4548
5	2 4 6 8 10 12 14	3.49 5.24 9.75 21.74 22.50 22.18 21.45 9.86	9.59 O. 21.79 O. 23.09 O. 24.08 O. 23.67 O. 22.74 O.	025 028 039 048 052 044 036	988 1068 1169 1304 1409 1415 1348 1258	980 1408 2620 4605 6465 5544 3863 2559
2.5	2 4 6 8 10 12 14	3.42 5.39 8.25 21.56 22.25 21.86 21.21 8.15	9.89 O. 21.48 O. 22.87 O. 23.75 O. 23.26 O. 22.44 O.	024 028 033 042 046 040 032 026	994 1077 1168 1316 1410 1391 1341 1240	960 1445 2207 4153 5730 4759 3373 2118

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames		Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.30 4.66 8.57 21.69 22.30 21.96 21.31 8.14	4.89 8.54 21.55 23.02 23.81 23.38 22.55 21.46	0.019 0.026 0.036 0.048 0.051 0.045 0.036 0.029	973 1055 1153 1293 1382 1366 1319 1169	712 1255 2313 4484 5941 5042 3589 2145
15	2 4 6 8 10 12 14	2.35 4.75 8.78 21.75 22.21 22.03 21.29 7.88	4.96 8.68 21.60 23.10 23.70 23.47 22.53 21.40	0.019 0.027 0.038 0.049 0.048 0.045 0.036	977 1049 1144 1297 1388 1380 1316 1216	721 1281 2381 4625 5678 5211 3567 2063
10	2 4 6 8 10 12 14	2.68 4.86 8.98 21.67 22.18 22.06 21.23 7.88	5.42 8.89 21.64 23.01 23.66 23.51 22.45 21.40	0.020 0.027 0.038 0.048 0.046 0.045 0.034 0.026	989 1056 1152 1291 1398 1382 1323 1228	788 1309 2430 4457 5568 5282 3423 2051
5	2 4 6 8 10 12 14	3.01 5.08 8.70 21.66 22.15 21.92 13.13 8.74	5.91 9.28 21.58 23.00 23.62 23.32 22.27 21.60	0.021 0.028 0.036 0.046 0.044 0.042 0.032 0.028	1001 1059 1158 1304 1408 1383 1314 1240	860 1368 2344 4413 5460 4902 3152 2281
2.5	2 4 6 8 10 12 14 16	3.31 5.24 8.32 21.52 22.15 21.92 12.88 7.98	6.36 9.59 21.50 22.81 23.62 23.33 22.25 21.43	0.022 0.028 0.033 0.043 0.044 0.041 0.031	1011 1068 1171 1299 1408 1392 1321 1246	928 1408 2227 4075 5460 4887 3109 2067

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromet 1 flame		Emiss.	Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.65 4.41 7.58 21.44 13.69 10.61 8.02 5.49	5.39 8.12 21.33 22.73 22.33 21.96 21.44 10.14	0.019 0.021 0.025 0.030 0.026 0.024 0.021	1001 1101 1228 1412 1396 1359 1307 1203	778 1163 1968 3758 3129 2642 2031 1399
15	2 4 6 8 10 12 14	2.76 4.57 7.76 21.52 21.54 21.14 8.11 5.93	5.55 8.39 21.37 22.83 22.85 22.35 21.46 11.12	0.019 0.022 0.027 0.033 0.032 0.028 0.021	1010 1098 1211 1393 1408 1373 1311 1213	799 1207 2032 3949 3963 3193 2052 1515
10	2 4 6 8 10 12 14	2.89 4.85 7.95 21.57 21.83 21.36 9.11 6.42	5.74 8.88 21.42 22.89 23.22 22.62 21.68 12.43	0.020 0.024 0.029 0.036 0.031 0.023	1006 1090 1196 1372 1423 1383 1322 1225	831 1286 2099 4089 4622 3618 2311 1643
5	2 4 6 8 10 12 14 16	3.01 4.89 8.15 21.60 22.00 21.57 10.65 7.23	5.91 8.94 21.46 22.92 23.44 22.89 21.96 21.24	0.021 0.025 0.031 0.040 0.039 0.034 0.026	1001 1080 1184 1340 1426 1392 1332 1238	860 1302 2167 4208 5045 4061 2676 1862
2.5	2 4 6 8 10 12 14 16	3.31 5.04 8.26 21.59 22.01 21.63 12.30 7.46	6.36 9.22 21.49 22.91 23.45 22.96 22.19 21.30	0.022 0.026 0.032 0.042 0.041 0.036 0.029	1011 1078 1178 1322 1409 1383 1334 1249	928 1345 2202 4220 5097 4201 3003 1917

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame		Emiss.	Flame Temp.	Rad. (Kcal. -2h1 ster1)
20	2 4 6 8 10 12 14	2.72 4.83 10.16 21.89 22.48 22.71 22.42 12.51	8.83 21.87 23.28 24.05 24.39 23.95	0.021 0.026 0.038 0.048 0.053 0.051 0.063	979 1065 1190 1330 1399 1452 1327 1315	801 1295 2703 4921 6428 7018 6407 3059
15	2 4 6 8 10 12 14	2.94 4.93 9.70 21.76 22.45 22.54 21.88 11.90		0.022 0.026 0.037 0.046 0.051 0.048 0.040	983 1072 1184 1321 1408 1445 1394 1292	850 1320 2592 4613 6329 6495 4796 2977
10	2	3.20	6.18	0.023	990	909
	4	5.03	9.21	0.027	1067	1351
	6	9.00	21.65	0.035	1178	2409
	8	21.67	23.00	0.045	1312	4405
	10	22.33	23.86	0.049	1402	5988
	12	22.33	23.11	0.045	1433	5921
	14	21.64	22.97	0.037	1376	4244
	16	8.71	21.59	0.029	1228	2283
5	2	3.12	6.07	0.023	984	893
	4	5.03	9.21	0.027	1067	1351
	6	8.50	21.54	0.033	1178	2271
	8	21.57	22.88	0.043	1309	4184
	10	22.11	23.57	0.046	1385	5400
	12	21.92	23.33	0.041	1392	4897
	14	21.37	22.63	0.034	1352	3677
	16	9.78	21.81	0.028	1279	2521
2.5	2	3.18	6.16	0.022	1001	901
	4	4.94	9.05	0.026	1072	1323
	6	8.15	21.46	0.032	1173	2173
	8	21.48	22.76	0.042	1300	3988
	10	21.77	23.14	0.041	1364	4576
	12	21.76	22.13	0.039	1380	4525
	14	21.20	22.42	0.032	1339	3353
	16	7.72	21.37	0.023	1262	1983

Add. Conc. (%)	Ht.  above Burner (cm)	Pyrometo 1 flame	er Output 2 flames		Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal}}}{\frac{m^{-2}h.}{-1}}$
20	2 4 6 8 10 12 14	2.42 6.90 10.26 21.58 22.54 22.67 21.91 12.46	5.05 21.15 21.89 22.89 24.13 24.30 23.32 22.21	0.019 0.029 0.038 0.040 0.052 0.049 0.040	983 1148 1193 1336 1416 1460 1400 1303	734 1834 2727 4158 6564 6856 4861 3064
15	2 4 6 8 10 12 14 16	2.68 4.76 9.53 21.83 22.42 22.46 21.35 11.51	5.42 8.71 21.75 23.21 23.97 24.03 22.61 22.09	0.020 0.025 0.036 0.045 0.050 0.046 0.031 0.032	989 1072 1188 1342 1409 1448 1380 1283	788 1272 2544 4753 6216 6259 3597 2911
10	2 4 6 8 10 12 14 16	2.99 5.14 9.15 21.69 22.22 22.24 21.53 9.60	5.88 9.41 21.68 23.03 23.72 24.06 22.83 21.77	0.021 0.026 0.035 0.044 0.047 0.044 0.036 0.028	1000 1084 1184 1324 1398 1425 1364 1272	855 1371 2446 4448 5689 5682 4010 2480
5	2 4 6 8 10 12 14	3.03 5.06 8.36 21.59 22.09 21.69 21.23 9.19	5.94 9.26 21.51 22.91 23.55 23.04 22.46 21.70	0.022 0.026 0.032 0.042 0.043 0.039 0.032 0.026	990 1080 1182 1321 1406 1367 1344 1283	869 1351 2227 4212 5306 4378 3400 2365
2.5	2 4 6 8 10 12 14	3.31 4.94 7.78 21.46 22.05 21.70 13.36 7.91	6.36 9.05 21.37 22.75 23.50 23.05 22.30 21.41	0.022 0.026 0.031 0.041 0.043 0.038 0.031	1011 1072 1168 1305 1400 1377 1328 1256	928 1323 2073 3949 5225 4375 3165 2039

Add. Conc. (%)	Ht. above Burner (cm)	Pyromete 1 flame	er Output 2 flames	Emiss.	Flame Temp.	<u>Kcal.</u> <u>m<sup>-2</sup>h1</u> <u>ster1</u> )
20	2 4 6 8 10 12 14 16	2.36 4.20 6.91 21.15 21.53 21.25 8.34 5.84	4.97 7.76 21.16 22.35 22.84 22.49 21.51 10.90	0.019 0.023 0.028 0.032 0.031 0.029 0.021	978 1060 1159 1327 1418 1384 1321 1208	723 1129 1827 3255 3935 3398 2106 1492
15	2 4 6 8 10 12 14 16	2.63 4.33 7.48 21.28 21.89 21.43 9.26 6.33	5.35 7.97 21.30 22.51 23.30 22.71 21.71 12.17	0.020 0.024 0.030 0.035 0.042 0.032 0.023	985 1056 1164 1323 1379 1386 1328 1220	778 1164 1989 3524 4584 3764 2344 1622
10	2 4 6 8 10 12 14 16	2.76 4.61 7.02 21.38 21.94 21.59 10.35 7.09	5.54 8.45 21.18 22.64 23.36 22.91 21.92 21.21	0.020 0.025 0.029 0.037 0.038 0.034 0.025	996 1063 1153 1324 1424 1396 1336 1231	804 1236 1863 3740 4898 4092 2604 1826
5	2 4 6 8 10 12 14 16	3.06 4.85 7.78 21.42 22.01 21.67 12.12 6.74	5.99 8.88 21.37 22.69 23.45 23.02 22.17 13.63	0.021 0.026 0.031 0.039 0.040 0.036 0.028	1005 1067 1168 1313 1418 1392 1343 1243	870 1301 2073 3833 5078 4291 2963 1721
2.5	2 4 6 8 10 12 14 16	3.15 4.85 8.18 21.50 22.05 21.71 11.76 7.09	6.12 8.88 21.47 22.79 23.50 23.07 22.11 21.21	0.022 0.026 0.032 0.041 0.042 0.038 0.045 0.023	999 1067 1175 1312 1408 1380 1176 1231	5202 4409 3084

Add. Conc. (%)	Ht. above Burner	Pyromet 1 flame		Emiss.	Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{1}{m^{-2}h}} - 1$
	(cm)					ster1)
20	2	2.29		0.019	972	710
	4	4.24		0.024	1051	1145
	6 8	6.40		0.026	1156	1682
	10	12.30 21.63		0.031	1311 1415	3033 4158
	12	21.05	- 11 H. 14 H. 14 H. 17 H. 17 H. 17 H.	0.028	1396	3376
	14	8.88		0.021	1344	2231
	16	6.50		0.020	1246	1653
15	2	2.43		0.020	969	740
	4	4.46		0.025	1053	1201
	6	6.56		0.027	1152	1731
	8	12.68		0.032	1307	3095
	10 12	21.64 21.37		0.034	1407	4203
	14	9.65		0.030	1398 1342	3632 2429
	16	8.16		0.033	1176	2261
10	2	2.55	5.23	0.020	979	763
	4	4.51		0.025	1056	1212
	6	6.91		0.028	1159	1827
	8	21.18		0.034	1312	3335
	10	21.77		0.036	1411	4493
	12	21.43		0.032	1386	3764
	14	10.35		0.025	1336	2604
	16	7.36		0.023	1244	1893
5	2	2.87		0.021	991	832
	4	4.66		0.026	1055	1255
	6	7.50		0.030	1165	1993
	8	21.28		0.037	1304	3557
	10	22.24		0.047	1400	5722
	12	21.56		0.035	1380	4053
	14	11.90		0.028	1338	2928
	16	7.82	21.39	0.024	1252	2018
2.5	2	3.17	6.15	0.022	1000	898
	4	5.09		0.026	1081	1358
	6	7.71		.030	1175	2047
	8	21.34		0.039	1299	3696
	10	21.97		0.041	1401	5001
	12	21.63	이전시 보기 이 집에 생겨면서	0.037	1374	4227
	14	12.91		0.030	1333	3100
	16	8.17	21.47	0.026	1241	2122

Add. Conc. (%)	Ht. above Burner (cm)	Pyromete 1 flame	er Output E 2 flames	miss.	Flame Temp.	Rad. (Kcal. -2h1 ster1)
20	2 4 6 8 10 12 14	2.24 4.18 6.90 21.25 21.71 21.41 11.76 5.81	7.73 0. 21.15 0. 22.47 0. 23.06 0. 22.68 0. 22.13 0.	019 024 029 039 041 036 030	968 1047 1148 1280 1353 1340 1311 1206	701 1131 1834 3519 4452 3780 2931 1486
15	2 4 6 8 10 12 14 16	2.37 4.29 7.10 21.25 21.77 21.51 11.49 7.01	7.91 0. 21.21 0. 22.47 0. 23.14 0. 22.81 0. 22.09 0.	019 024 029 039 041 037 030 024	979 1054 1157 1280 1364 1350 1304 1213	724 1156 1884 3519 4576 3986 2884 1818
10	2 4 6 8 10 12 14	2.71 4.42 7.58 21.33 21.89 21.57 12.47 6.91	8.13 O. 21.33 O. 22.57 O. 23.30 O. 22.89 O. 22.21 O.	020 025 030 040 042 037 031 023	992 1051 1169 1287 1379 1363 1314 1222	794 1192 2015 3676 4584 4113 3053 1783
5	2 4 6 8 10 12 14 16	2.96 4.42 7.80 21.38 21.91 21.58 12.74 6.70	8.13 0. 21.38 0. 22.64 0. 23.31 0. 22.90 0. 22.24 0.	021 025 030 041 042 038 031	997 1051 1179 1289 1381 1355 1319 1211	849 1192 2069 3791 4883 4142 3090 1731
2.5	2 4 6 8 10 12 14 16	3.31 4.57 7.73 21.43 22.01 21.66 21.14 7.49	8.38 0.0 21.36 0.0 22.70 0.0 23.45 0.0 23.00 0.0 22.35 0.0	022 025 031 041 043 038	1011 1060 1165 1299 1392 1370 1337 1224	928 1227 2059 3886 5136 4299 3236 1947

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames		Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m-2}}{\text{h.}}-1}$ $\frac{\text{ster.}^{-1}}{}$
20	2 4 6 8 10 12 14	2.70 5.57 21.32 22.61 23.67 24.26 23.86 22.51	5.46 10.24 22.54 24.17 25.54 26.33 25.87 24.02	0.022 0.032 0.060 0.088 0.116 0.134 0.119 0.098	965 1048 1152 1244 1297 1320 1313 1193	803 1515 3846 7228 10948 13411 11696 7032
15	2 4 6 8 10 12 14	3.16 5.57 21.24 22.50 23.67 24.10 22.54 22.26	6.13 10.24 22.44 24.04 25.57 26.14 24.12 23.72	0.024 0.032 0.057 0.080 0.106 0.118 0.057 0.084	976 1048 1154 1259 1329 1346 1382 1205	906 1515 3670 6839 10845 12587 6639 6227
10	2 4 6 8 10 12 14	3.49 6.12 13.49 22.61 23.68 23.62 22.77 21.72	6.62 11.50 22.28 24.18 25.60 25.52 24.39 23.05	0.025 0.034 0.052 0.078 0.097 0.094 0.075 0.061	988 1060 1159 1284 1362 1364 1321 1219	980 1669 3393 7110 10762 10491 7521 4691
5	2 4 6 8 10 12 14	3.77 5.78 11.15 22.27 23.21 22.59 22.04 21.18	7.06 10.70 22.03 23.76 24.99 24.19 23.47 22.39	0.026 0.031 0.044 0.064 0.072 0.058 0.050 0.042	996 1069 1170 1297 1403 1385 1343 1242	1048 1562 2962 6040 8816 6808 5292 3435
2.5	2 4 6 8 10 12 14 16	3.51 5.54 10.19 21.83 22.83 22.38 21.65 10.10	6.65 10.21 21.87 23.21 24.50 23.92 22.99 21.87	0.024 0.028 0.039 0.048 0.059 0.051 0.040	1000 1086 1183 1320 1417 1396 1350 1255	980 1483 2720 4804 7476 6138 4309 2622

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m-2}}{\text{h.}}-1}$ $\frac{\text{ster.}^{-1}}{}$
20	2 4 6 8 10 12 14	2.40 4.84 21.34 22.68 23.63 24.12 23.45 22.31	8.86 22.58 24.28 25.56 26.25 25.32	0.020 0.026 0.055 0.070 0.080 0.086 0.079	967 1066 1185 1332 1425 1465 1404 1299	734 1298 3861 7205 10330 12189 9710 6160
15	2 4 6 8 10 12 14	2.65 5.01 21.30 22.42 23.31 23.83 23.01 21.90	9.16 22.52 23.95 25.13 25.83 24.72	0.021 0.027 0.054 0.063 0.072 0.081 0.071	974 1065 1183 1327 1418 1448 1377 1272	789 1345 3766 6407 9140 11022 8175 5048
10	2 4 6 8 10 12 14	3.06 5.20 11.76 22.06 23.17 22.96 22.29 21.32	9.52 22.11 23.50 24.95 24.67 23.80	0.023 0.028 0.045 0.014 0.069 0.061 0.053	980 1066 1176 1320 1413 1425 1367 1277	879 1398 3084 5394 8661 7877 5950 3676
5	2 4 6 8 10 12 14	3.53 5.46 9.46 21.65 22.36 22.06 21.38 9.02	10.03 21.74 22.99 23.89 23.51 22.65	0.025 0.029 0.037 0.044 0.049 0.042 0.034 0.027	991 1071 1176 1317 1407 1409 1355 1264	990 1468 2535 4368 6057 5222 3706 2337
2,5	2 4 6 8 10 12 14 16	3.53 5.25 8.05 21.55 22.16 21.86 21.21 8.71	9.62 21.44 22.85 23.63 23.26 22.44	0.024 0.027 0.032 0.042 0.045 0.040 0.032 0.029	1002 1080 1169 1313 1402 1391 1341 1228	985 1403 2150 4128 5500 4759 3373 2283

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromet 1 flame		t Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}}$
20	2 4 6 8 10 12 14	2.47 4.94 21.38 22.74 23.81 24.37 23.64 22.60	5.12 9.04 22.62 24.36 25.80 26.59 25.56 24.18	0.021 0.027 0.056 0.074 0.087 0.096 0.088 0.074	960 1061 1185 1321 1419 1456 1392 1300	753 1329 3931 7421 11066 13308 10490 7041
15	2 4 6 8 10 12 14	2.76 5.01 21.30 22.87 23.45 23.85 23.08 22.30	5.53 9.16 22.52 24.53 25.32 25.87 24.81 23.79	0.022 0.027 0.054 0.079 0.077 0.082 0.073 0.063	969 1065 1182 1319 1414 1447 1378 1307	813 1345 3758 7875 9683 11137 8421 6093
10	2 4 6 8 10 12 14	3.14 5.21 12.74 22.10 23.73 23.19 22.31 21.57	6.09 9.54 22.22 23.55 25.67 24.99 23.83 22.88	0.024 0.028 0.047 0.056 0.092 0.063 0.054 0.046	974 1067 1180 1314 1387 1450 1364 1287	901 1401 3256 5515 10841 8621 6015 4228
5	2 4 6 8 10 12 14	3.43 5.53 9.84 21.66 22.49 22.19 21.53 9.73	6.53 10.17 21.81 22.99 24.06 23.69 22.84 21.80	0.025 0.029 0.038 0.045 0.051 0.044 0.038 0.029	984 1075 1180 1311 1415 1417 1345 1265	968 1486 2633 4396 6426 5575 4046 2521
2.5	2 4 6 8 10 12 14 16	3.36 4.99 7.92 21.53 22.23 21.94 21.19 7.41	6.43 9.13 21.41 22.83 23.73 23.36 22.40 21.29	0.023 0.027 0.032 0.042 0.046 0.040 0.032 0.023	1002 1064 1164 1310 1407 1405 1336 1247	944 1341 2116 4095 5687 4926 3327 1905

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flame		Flame Temp.	Rad. (Kcal. m <sup>-2</sup> h. <sup>-1</sup> ster. <sup>-1</sup> )
20	2 4 6 8 10 12 14	2.40 5.05 21.61 22.96 24.10 24.73 23.93 22.85	5.02 9.23 22.90 24.64 26.18 27.08 25.94 24.49	0.020 0.028 0.062 0.084 0.100 0.110 0.098 0.085	967 1057 1194 1312 1405 1448 1391 1291	734 1361 4459 8223 12315 14968 11660 7893
15	2 4 6 8 10 12 14	2.87 5.25 21.47 22.50 23.76 24.14 23.39 22.38	5.63 9.61 22.74 24.06 25.72 26.26 25.22 23.89	0.022 0.029 0.057 0.067 0.087 0.093 0.083	974 1059 1197 1319 1412 1438 1377 1288	826 1417 4135 6679 10878 12375 9556 6368
10	2 4 6 8 10 12 14	3.22 5.45 13.06 22.18 23.23 23.03 22.48 21.64	6.21 10.00 22.25 23.65 25.02 24.76 24.04 22.96	0.024 0.030 0.047 0.057 0.072 0.068 0.059 0.049	980 1060 1185 1322 1406 1397 1360 1277	917 1473 3300 5728 8884 8216 6521 4394
5	2 4 6 8 10 12 14	3.70 5.40 9.63 21.79 22.48 22.14 21.51 10.36	6.95 9.91 21.77 23.16 24.04 23.61 22.80 21.91	0.026 0.029 0.037 0.048 0.053 0.045 0.037	992 1068 1182 1313 1398 1399 1349 1263	1033 1454 2575 4718 6416 5458 3978 2678
2.5	2 4 6 8 10 12 14 16	3.57 5.15 8.69 21.56 22.22 21.81 21.29 8.90	6.76 9.43 21.58 22.87 23.72 23.20 22.53 21.63	0.024 0.027 0.033 0.043 0.046 0.040 0.033	1005 1074 1185 1308 1406 1381 1345 1272	995 1380 2317 4167 5676 4650 3513 2298

Add. Conc. (%)	Burner (cm)	Pyrometo 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{-2}{\text{h}}} - 1$ $\frac{\text{ster}}{}$
20	2 4 6 8 10 12 14	2.51 4.82 21.35 22.63 23.51 23.98 23.34 22.33	8.83 22.59 24.22 25.41 26.05 25.17	0.020 0.025 0.053 0.066 0.075 0.080 0.073	976 1076 1198 1344 1432 1473 1417	755 1287 3853 7013 9850 11548 9250 6142
15	2 4 6 8 10 12 14	2.71 4.66 21.29 22.36 23.25 23.87 23.03 22.15	8.54 22.51 23.88 25.06 25.91 24.76	0.022 0.026 0.052 0.060 0.070 0.078 0.068 0.059	966 1055 1192 1335 1420 1468 1397 1305	805 1255 3723 6225 8920 11136 8216 5683
10	2 4 6 8 10 12 14 16	3.18 4.82 12.21 22.02 22.97 23.11 22.23 21.28	8.82 22.17 23.44 24.69 24.88 23.73	0.023 0.027 0.044 0.052 0.063 0.059 0.050 0.040	988 1054 1192 1325 1416 1461 1377 1278	904 1300 3150 5267 7953 8285 5757 3594
5	2 4 6 8 10 12 14 16	3.51 5.12 7.64 21.67 22.35 22.09 10.61 8.52	9.36 21.34 23.01 23.89 23.55 21.96	0.024 0.028 0.027 0.043 0.049 0.040 0.024	1000 1061 1205 1328 1406 1433 1359 1269	980 1378 2002 4391 6046 5263 2642 2196
2.5	2 4 6 8 10 12 14 16	3.41 5.06 8.26 21.59 22.18 21.82 21.23 7.79	9.26 21.49 22.91 23.66 23.21	0.023 0.027 0.032 0.042 0.045 0.039 0.032 0.024	1005 1068 1178 1322 1406 1392 1345 1251	953 1357 2202 4220 5552 4658 3407 2009

Add. Conc. (%)	Ht. above Burner (cm)	Pyrometo 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{m^{-2}h.}{-1}}$
20	2 4 6 8 10 12 14 16	2.47 6.42 8.93 21.81 22.43 22.08 21.57 10.21	12.43 21.63 23.18 23.99 23.53 22.89	0.020 0.021 0.036 0.049 0.052 0.046 0.040	972 1225 1167 1308 1398 1380 1335 1228	747 1643 2402 4758 6294 5337 4150 2679
15	2 4 6 8 10 12 14	2.77 5.36 8.38 21.70 22.24 21.96 21.46 9.80	9.83 21.51 23.04 23.73 23.38 22.75	0.021 0.029 0.034 0.046 0.049 0.043 0.037	983 1065 1164 1311 1385 1382 1340 1245	811 1444 2249 4494 5752 5009 3893 2557
10	2 4 6 8 10 12 14 16	2.69 5.43 8.46 21.57 22.24 21.86 21.29 9.11	9.98 21.53 22.87 23.74 23.25 22.54	0.021 0.028 0.033 0.044 0.047 0.041 0.034 0.028	976 1080 1176 1300 1400 1380 1336 1255	795 1455 2261 4187 5722 4757 3542 2369
<b>.</b>	2 4 6 8 10 12 14	3.02 4.97 8.26 21.47 22.13 21.82 21.15 8.20	9.08 21.49 22.76 23.60 23.20 22.36	0.022 0.027 0.032 0.042 0.045 0.039 0.032	989 1063 1178 1299 1398 1392 1328 1243	867 1335 2202 3980 5447 4649 3268 2131
2.5	2 4 6 8 10 12 14	3.22 5.03 8.05 21.52 22.33 21.76 21.16 7.07	9.21 21.42 22.82 23.11 23.13 22.37	0.022 0.027 0.032 0.042 0.045 0.038 0.031	1004 1067 1169 1308 1433 1389 1340 1230	907 1351 2150 4070 5921 4504 3261 1822

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromet 1 flame		Emiss.	Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.45 4.39 8.05 21.52 22.16 21.87 21.24 7.76	5.10 8.07 21.44 22.82 23.64 23.26 22.47 21.37	0.019 0.024 0.032 0.043 0.047 0.042 0.035 0.027	985 1060 1169 1300 1388 1374 1316 1211	739 1178 2150 4092 5549 4798 3461 2032
15	2 4 6 8 10 12 14	2.75 4.53 8.42 21.43 22.17 21.78 21.24 8.03	5.53 8.32 21.52 22.70 23.65 23.16 22.47 21.44	0.020 0.025 0.032 0.042 0.046 0.041 0.034 0.027	995 1058 1184 1291 1396 1367 1325 1223	802 1218 2242 3900 5547 4603 3444 2099
10	2 4 6 8 10 12 14	3.05 4.64 7.96 21.52 22.19 21.79 21.24 7.87	5.97 8.49 21.42 22.82 23.68 23.17 22.47 21.40	0.021 0.026 0.032 0.042 0.045 0.040 0.033	1004 1053 1165 1308 1408 1377 1336 1241	868 1249 2126 4078 5584 4605 3431 2041
5	2 4 6 8 10 12 14	3.12 4.83 7.89 21.45 22.09 21.76 21.16 7.65	6.07 8.83 21.40 22.73 23.55 23.13 22.37 21.35	0.022 0.026 0.031 0.042 0.044 0.039 0.032 0.025	996 1065 1172 1294 1398 1380 1330 1231	887 1295 2101 3932 5326 4525 3281 1985
2.5	2 4 6 8 10 12 14 16	3.24 4.75 8.00 21.50 22.12 21.74 12.79 7.16	6.25 8.69 21.43 22.79 23.59 23.10 22.24 21.23	0.022 0.026 0.031 0.041 0.043 0.038 0.031	1005 1060 1177 1312 1412 1385 1320 1249	912 1276 2129 4014 5377 4461 3096 1834

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output Emiss. 2 flames	$\frac{\frac{\text{Flame}}{\text{Temp.}}}{\binom{\text{OC}}{}} \frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{-2}{\text{h.}}-1}$ $\frac{\text{ster.}^{-1}}{})$
20	2 4 6 8 10 12 14	2.35 7.52 21.31 22.63 23.70 23.91 23.20 22.22	4.95 0.020 21.30 0.044 22.53 0.061 24.20 0.079 25.63 0.095 25.89 0.110 24.94 0.095 23.67 0.080	963 724 1051 2098 1147 3849 1282 7172 1372 10790 1347 11757 1303 9095 1215 6086
15	2 4 6 8 10 12 14	2.83 7.51 11.90 22.43 23.50 23.42 22.72 21.75	5.64 0.022 21.30 0.043 22.14 0.032 23.95 0.073 25.37 0.087 25.25 0.093 24.33 0.077 23.08 0.064	975 828 1057 2090 1292 2977 1278 6559 1376 9997 1341 9803 1304 7402 1208 4786
10	2 4 6 8 10 12 14	3.30 7.19 21.14 22.31 23.32 22.92 22.13 21.33	6.34 0.024 21.22 0.040 22.33 0.052 23.80 0.067 25.09 0.078 24.61 0.074 23.59 0.059 22.57 0.049	986 936 1064 1987 1165 3454 1288 6170 1385 9156 1350 7972 1302 5637 1220 3776
5	2 4 6 8 10 12 14	3.73 6.15 10.73 21.87 22.69 22.17 21.63 10.96	6.99 0.025 11.58 0.034 21.96 0.044 23.25 0.053 24.32 0.059 23.64 0.051 22.96 0.044 22.01 0.036	1004 1034 1062 1677 1160 2878 1292 4941 1395 7087 1359 5614 1313 4325 1230 2852
2.5	2 4 6 8 10 12 14	3.51 5.46 9.15 12.51 22.42 21.85 21.22 8.36	6.65 0.024 10.03 0.029 21.68 0.037 22.27 0.031 23.97 0.050 23.24 0.042 22.45 0.034 21.51 0.028	1000 980 1071 1468 1166 2463 1315 3059 1409 6216 1370 4752 1322 3417 1225 2190

Add. Conc. (%)	Ht.  above Burner (cm)	Pyromet 1 flame	er Outpu 2 flame		Flame Temp.	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.43 5.97 9.95 21.99 22.71 22.21 21.59 8.12	5.06 11.12 21.83 23.40 24.34 23.70 22.90 21.46	0.020 0.034 0.042 0.057 0.063 0.052 0.043 0.029	969 1052 1152 1289 1375 1360 1312 1204	740 1629 2692 5271 7211 5736 4218 2141
15	2 4 6 8 10 12 14	2.72 5.72 8.85 21.84 22.47 22.06 21.41 8.43	5.48 10.55 21.61 23.21 24.02 23.50 22.68 21.53	0.021 0.032 0.038 0.052 0.056 0.048 0.040	979 1056 1147 1293 1376 1361 1303 1195	801 1552 2398 4858 6435 5315 3829 2234
10	2 4 6 8 10 12 14 16	2.85 5.41 8.35 21.79 22.29 21.89 21.38 8.52	5.68 9.92 21.50 23.06 23.80 23.29 22.64 21.55	0.021 0.030 0.035 0.048 0.051 0.045 0.038	989 1052 1163 1306 1386 1361 1321 1209	828 1462 2248 4549 5918 4896 3750 2248
5	2 4 6 8 10 12 14 16	2.90 5.14 8.13 21.64 22.16 21.80 21.27 8.31	5.75 9.40 21.46 22.96 23.63 23.18 22.50 21.50	0.022 0.029 0.033 0.045 0.047 0.042 0.035 0.028	980 1052 1163 1306 1386 1361 1321 1223	843 1390 2178 4343 5528 4651 3510 2176
2.5	2 4 6 8 10 12 14	3.11 5.15 8.18 21.62 22.13 21.74 21.17 8.35	6.05 9.41 21.47 22.94 23.59 23.10 22.38 21.51	0.022 0.028 0.032 0.043 0.045 0.040 0.032	996 1063 1175 1319 1396 1367 1332 1249	885 1384 2183 4286 5426 4490 3294 2167

Add. Conc. (%)	Ht. above Burner (cm)	Pyromete 1 flame	r Output 2 flames	Emiss.	Temp.	Rad. (Kcal. n <sup>-2</sup> h1 ster1)
20	2 4 6 8 10 12 14	2.53 5.09 8.72 21.86 22.50 22.23 21.62 9.53	5.20 9.31 21.59 23.24 24.07 23.73 22.94 21.76	0.020 0.028 0.036 0.049 0.054 0.049 0.042	977 1060 1159 1317 1395 1384 1326 1216	759 1371 2349 4865 6487 5741 4263 2516
15	2 4 6 8 10 12 14 16	2.50 5.16 8.08 21.67 22.32 22.03 21.38 8.91	5.16 9.44 21.44 23.00 23.84 23.47 22.65 21.63	0.020 0.028 0.034 0.046 0.050 0.045 0.047	975 1064 1152 1304 1392 1379 1325 1224	753 1388 2174 4422 5972 5201 3748 2342
10	2 6 8 10 12 14 16	2.75 4.95 7.99 21.56 22.26 21.94 21.26 8.84	5.52 9.06 21.42 22.86 23.76 23.36 22.50 21.62	0.021 0.027 0.033 0.044 0.048 0.042 0.034 0.028	981 1062 1157 1299 1396 1388 1330 1244	807 1332 2144 4170 5788 4959 3486 2305
5	2 4 6 8 10 12 14 16	3.00 4.92 7.88 21.52 22.08 21.77 21.44 8.18	5.89 9.00 21.40 22.81 23.53 23.14 22.71 21.47	0.022 0.026 0.032 0.042 0.045 0.040 0.032	988 1071 1162 1307 1388 1372 1326 1255	863 1317 2107 4062 5313 4552 3857 2115
2.5	2 6 8 10 12 14 16	4.71 4.88 7.78 21.49 22.12 21.72 21.13 7.87	8.60 8.93 21.37 22.78 23.58 23.08 22.33 21.40	0.022 0.026 0.031 0.042 0.044 0.039 0.031	999 1068 1168 1303 1403 1373 1335 1241	1300 1307 2073 4021 5388 4447 3216 2041

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss.	Flame Temp. (°C)	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.56 3.94 6.70 21.37 21.58 21.67 10.58 6.55	7.34 13.36 22.63 22.89 23.02 21.96	0.020 0.022 0.027 0.038 0.040 0.036 0.025	980 1056 1160 1312 1336 1392 1344 1248	765 1064 1766 3727 4158 4300 2651 1664
15	2 4 6 8 10 12 14 16	2.47 4.06 7.13 21.39 22.02 21.70 11.49 6.35	7.53 21.21 22.66 23.46 23.05 22.09	0.020 0.023 0.029 0.038 0.042 0.037 0.027	972 1051 1159 1317 1403 1388 1341 1237	747 1097 1892 3773 5143 4368 2846 1615
10	2 4 6 8 10 12 14	2.73 4.48 7.53 21.44 22.07 21.73 21.55 7.24	8.23 21.31 22.71 23.53 23.10 22.86	0.021 0.025 0.031 0.040 0.043 0.037 0.038 0.022	980 1055 1156 1308 1404 1394 1349 1253	803 1207 2009 3884 5275 4436 4086 1853
5	2 4 6 8 10 12 14 16	2.87 4.81 7.65 21.47 22.06 21.73 9.60 7.28	8.81 21.34 22.76 23.51 23.08 21.77	0.021 0.026 0.031 0.041 0.043 0.038 0.028 0.023	991 1064 1162 1307 1401 1383 1272 1240	832 1292 2041 3965 5245 4435 2480 1873
2.5	2 4 6 8 10 12 14	3.20 4.87 7.78 21.42 22.03 21.74 13.80 7.86	8.90 21.37	0.022 0.026 0.031 0.041 0.043 0.038 0.031	1003 1068 1168 1296 1395 1385 1334 1228	905 1304 2073 3862 5165 4461 3210 2047

Add. Conc. (%)	Ht.  above Burner (cm)	Pyrometo 1 flame	er Outpu 2 flame		Flame Temp. (°C)	Rad. (Kcal. m-2h1 ster1)
20	2 4 6 8 10 12 14	2.45 4.73 7.10 21.33 22.06 21.68 12.33 6.16	5.09 8.65 21.21 22.58 23.51 23.02 22.20 11.70	0.020 0.026 0.029 0.039 0.042 0.037 0.030 0.021	971 1059 1157 1296 1410 1383 1323 1211	743 1270 1884 3673 5232 4318 3021 1581
15	2 4 6 8 10 12 14 16	2.80 4.93 7.27 21.33 22.04 21.69 12.46 6.88	5.60 9.01 21.25 22.58 23.49 23.04 22.21 21.15	0.021 0.027 0.030 0.039 0.042 0.037 0.030	985 1060 1155 1296 1407 1386 1325 1220	817 1325 1936 3673 5192 4352 3039 1776
10	2 4 6 8 10 12 14	2.80 4.93 7.36 21.39 22.07 21.73 12.65 7.06	5.60 9.01 21.27 22.66 23.53 23.09 22.23 21.20	0.021 0.027 0.030 0.040 0.043 0.038 0.031	985 1060 1159 1300 1404 1384 1317 1216	817 1325 1958 3798 5275 4452 3078 1830
5	2 4 6 8 10 12 14	3.17 4.67 7.78 21.54 22.01 21.70 21.15 7.89	6.15 8.56 21.37 22.84 23.45 23.05 22.35 21.41	0.022 0.026 0.031 0.041 0.043 0.038 0.032 0.026	1000 1056 1168 1320 1392 1377 1327 1229	898 1258 2073 4095 5138 4375 3255 2056
2.5	2 4 6 8 10 12 14 16	3.17 4.85 7.78 21.49 22.03 21.73 13.80 7.82	6.15 8.88 21.37 22.78 23.47 23.09 22.33 21.39	0.022 0.026 0.031 0.041 0.043 0.038 0.031 0.025	1000 1067 1168 1311 1395 1384 1334 1239	898 1301 2073 4006 5165 4452 3210 2028

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame			Flame Temp.	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m-2}}{\text{h.}}-1}$ ster1
40	2	2.50	5.16	0.020	975	753
	4	4.46	8.19	0.025 0.029	1053 1148	1201 1838
	6	6.92	21.16			3696
	8	21.34	22.60	0.039	1299	
	10	21.98	23.40	0.041	1403	5020
	12	21.67	23.02	0.036	1392	4291
	14	11.98	22.15	0.030	1316	2967
	16	5.92	11.07	0.021	1196	1520

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Output 2 flames	Emiss.	Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{\text{m}^{-2}\text{h}}{\text{-1}}}$
20	2 4 6 8 10 12 14	2.66 4.72 7.50 21.52 22.11 21.74 21.24 7.25	5.39 8.65 21.31 22.82 23.58 23.10 22.47 21.55	0.020 0.025 0.030 0.041 0.045 0.041 0.034 0.024	988 1070 1165 1316 1394 1358 1325 1225	784 1263 1993 4062 5395 4505 3444 1877
15	2	2.91	5.77	0.021	994	840
	4	4.72	8.65	0.025	1070	1263
	6	7.65	21.34	0.031	1162	2041
	8	21.54	22.84	0.041	1320	4095
	10	22.13	23.60	0.045	1397	5437
	12	21.77	23.14	0.042	1356	4596
	14	21.32	22.56	0.036	1321	3610
	16	7.49	21.31	0.026	1211	1957
10	2	3.01	5.91	0.021	1001	860
	4	4.65	8.51	0.025	1065	1245
	6	7.82	21.38	0.031	1169	2082
	8	21.52	22.82	0.042	1308	4070
	10	22.21	23.71	0.046	1404	5654
	12	21.82	23.20	0.042	1364	4687
	14	21.28	22.52	0.036	1314	3546
	16	7.70	21.36	0.026	1220	2008
5	2	3.23	6.23	0.022	1004	910
	4	5.04	9.22	0.026	1078	1345
	6	7.76	21.37	0.031	1167	2069
	8	21.60	22.92	0.042	1324	4237
	10	22.10	23.56	0.046	1384	5389
	12	21.85	23.24	0.043	1363	4780
	14	21.31	22.56	0.037	1311	3615
	16	7.99	21.43	0.028	1209	2098
2.5	2 4 6 8 10 12 14	3.41 5.12 8.18 21.64 22.15 21.82 21.27 7.99	6.51 9.36 21.47 22.97 23.63 23.20 22.51 21.43	0.022 0.026 0.032 0.042 0.046 0.043 0.037	1018 1083 1175 1331 1393 1357 1303 1209	949 1364 2183 4314 5505 4715 3542 2098

Add. Conc. (%)	Ht. above Burner (cm)	Pyromete 1 flame	er Output 2 flames	Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{\text{m}^{-2}\text{h}}{\text{m}^{-1}}}$
20	2	3.40	6.50	0.022	1017	947
-	4	10.33	21.90	0.039	1187	2750
	6	21.72	23.06	0.049	1292	4658
	8	12.74	22.23	0.036	1268	3149
	10	4.76	8.73	0.022	1111	1253
	12	2.78	5.58	0.018	1027	798
0	2	4.29	7.91	0.024	1054	1156
Ŭ	4	11.08	22.02	0.039	1207	2904
	6	21.86	23.24	0.050	1311	4885
	8	21.23	22.45	0.038	1284	3471
	10	5.00	9.17	0.022	1126	1309
	12	2.80	5.60	0.020	999	813

Add.	Ht. above Burner	Pyrometer Output Emiss.  1 flame 2 flames			Flame Temp.	Rad. (Kcal.
(%)	(cm)				<u>(°c)</u>	ster1)
20	2	5.57	10.20	0.039	992	1553
	4	21.83	23.22	0.081	1154	5215
	6	23.23	24.95	0.116	1240	9448
	8	22.71	24.28	0.104	1204	7677
	10	21.58	22.84	0.090	1075	4612
	12	10.18	21.84	0.074	992	2946
0	2	4.18=	7.73	0.024	1047	1131
<i>50</i>	4	10.93	22.00	0.040	1195	2883
	6	21.83	23.20	0.051	1299	4833
	8	21.18	22.39	0.039	1266	3397
	10	5.09	9.33	0.022	1132	1330
	12	2.83	5.65	0.019	1015	814

Additive sulphur trioxide on town gas after passing through silica gel and carbosorb

Add. Conc. (%)	Ht. above Burner (cm)	Pyrome: 1 flam	ter Outpu e 2 flame	it Emiss.	Flame Temp. (°C)	$\frac{\frac{\text{Rad.}}{(\text{Kcal.}}}{\frac{\text{m}^{-2}\text{h.}^{-1}}{\text{ster.}^{-1}}}$
20	2	3.64	6.86	0.024	1010	1010
	4	10.71	21.96	0.041	1181	2847
	6	21.81	23.18	0.053	1283	4821
	8	13.54	22.30	0.039	1252	3279
	10	4.70	8.62	0.027	1107	1238
	12	2.97	5.85	0.020	1012	846
0	2	4.23	7.81	0.024	1050	1142
	4	11.25	22.04	0.040	1203	2946
	6	21.82	23.19	0.050	1303	4787
	8	21.20	22.41	0.040	1262	3448
	10	4.79	8.79	0.021	1127	1252
	12	2.83	5.65	0.020	1001	819

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame			Flame Temp. (°C)	$\frac{\frac{\text{Rad}}{(\text{Kcal})}}{\frac{\text{m}^{-2}\text{h}}{\text{ster}}}$
20	2	5.56	10.20	0.037	1007	1541
	4	21.91	23.27	0.080	1164	5303
	6	23.34	25.10	0.116	1255	9813
	8	22.72	24.28	0.106	1198	7706
	10	21.57	22.82	0.092	1067	4603
	12	11.11	21.99	0.076	1007	3166
0	2	4.18	7.73	0.024	1047	1131
	4	10.93	22.00	0.040	1195	2883
	6	21.83	23.20	0.050	1305	4816
	8	21.20	22.41	0.039	1269	3425
	10	5.09	9.33	0.022	1132	1330
	12	2.97	5.85	0.021	998	851

Add. Conc. (%)	Burner (cm)	Pyromet 1 flame	er Outpu 2 flame		Flame Temp.	Rad. (Kcal. in -2 h1
						ster. )
20	2	3.71	6.97	0.024	1015	1025
	4	10.71	21.96	0.040	1189	2839
	6	21.78	23.14	0.053	1277	4752
	8	21.19	22.39	0.040	1259	3419
	10	4.70	8.62	0.022	1107	1238
	12	3.14	6.10	0.021	1011	886
0	2	4.18	7.73	0.024	1047	1131
	4	10.93	22.00	0.040	1195	2883
	6	21.83	23.20	0.051	1299	4833
	8	21.19	22.39	0.040	1259	3419
	10	5.09	9.33	0.022	1132	1330
	12	2.79	5.59	0.020	998	810

Run No. G31

Additive sulphur trioxide and tri-ethylene glycol on town gas passed through silica gel

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame			Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{m^{-2}h}{-1}}$ $\frac{\text{ster.}^{-1}}{)$
20% SO <sub>2</sub>	2	3.67	6.90	0.024	1012	1015
3	4	10.67	21.95	0.041	1180	2841
	6	21.84	23.21	0.054	1281	4892
	8	21.27	22.50	0.044	1244	3622
	10	4.80	8.78	0.024	1087	1274
	12	3.33	6.39	0.021	1025	926

Run No. G32

Additive sulphur trioxide and tetrahydrothiophene on town gas, passed through silica gel

Add. Ht.  Conc. above Burner (cm)	Pyromete 1 flame	2 flames	Emiss.	Flame Temp. (°C)	<u>(Kcal</u> . <u>-2</u> <u>ster1</u>
20% SO <sub>2</sub> 2	5.62	10.33	0.037	1010	1557
3 4	21.84	23.19	0.077	1165	5115
6	23.11	24.80	0.108	1247	8946
8	22.67	24.24	0.098	1217	7504
10	21.55	22.80	0.084	1090	4502
12	10.29	21.86	0.069	1015	2947

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame			Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{m^{-2}h}{\text{h.}}}$
	2	2.90	5.75	0.021	993	838
	4	4.94	9.04	0.027	1061	1329
	6	8.11	21.45	0.033	1162	2173
	8	21.58	22.89	0.043	1311	4201
	10	22.08	23.53	0.045	1388	5313
	12	21.72	23.08	0.039	1372	4438
	14	21.14	22.34	0.033	1315	3257
	16	7.58	21.33	0.026	1215	1978

Add. Conc. (%)	Ht. above Burner (cm)	Pyromet 1 flame	er Outpu		Flame Temp.	$\frac{\frac{\text{Rad}}{(\text{Kcal}}}{\frac{\text{m}^{-2}\text{h}}{\text{-1}}}$
						ster. )
20% SO	2	3.69	6.92	0.027	980	1035
	3 4	8.20	21.46	0.049	1047	2309
	6	21.29	22.50	0.060	1148	3794
	8	22.48	24.02	0.071	1296	6674
	10	23.21	24.99	0.080	1365	8946
	12	22.73	24.35	0.069	1344	7331
	14	21.48	22.76	0.044	1284	4011
	16	7.96	21.42	0.030	1186	2111

Appendix 7
Table 16

The visual effects of the addition of sulphur trioxide.

			,	•					
.5	conclusion	may need richer initial flame	may need SO <sub>3</sub> in fuel supply	check earlier with town gas	may be due to $c_2H_6$ or $c_3H_8$ in town gas	luminosity not due to $c_2^{\rm H}6$	luminosity not due to $c_{5}^{\rm H}$ 8	luminosity due to a trace component	trace component removed by carbosorb or silica gel
The visual effects of the addition of sulphur trioxide.	visual effect	none	none	none	strongly luminous throughout flame	none	none	none	none
addition o	other additive after absorber	ou	ou	ou	on	ou	ou .	ou	ou
s of the	fuel through carbo- silica sorb gel	yes	yes	yes	ou	yes	yes	yes	yes
effect	fuel the	yes	yes	yes	ou	yes	yes	yes	yes
The visual	other additive before absorber	no	ou	ou	ou	$c_{2}^{\mathrm{H}}$	$^{\mathrm{c}_{3^{\mathrm{H}}8}}$	ou	ou
•	SO <sub>3</sub> added to	afr	air	CH <sub>4</sub>	air	afr	air	air	air
	Fuel	a. CH4	b. rich CH	c. rich	d. town	e. CH <sub>4</sub>	f. CH4	g. sim. town gas	h. town gas

	conclusion	trace component absorbed by silica gel	trace component not absorbed by carbo- sorb	trace component not H <sub>2</sub> 0		trace component not t.e.g.		trace component not $c_{\rm H}_6$	· .
	visual effect	none	strongly luminous	none	sl. yellow tip	sl. yellow tip	sl. yellow tip	sl. yellow tip	sl. yellow tip
	other additive after absorber	ou .	ou	н <sub>2</sub> 0	t.e.g.	ڻ 6 8	9 <sub>H</sub> 9 <sub>2</sub>	9 <sub>H</sub> 9 <sub>2</sub>	t,h.t.
,	hrough silica gel	yes	· ou	yes	yes	yes	yes	yes	yes
	fuel through carbo- sili sorb gel	ou .	yes	ou	no	ou	ou	no	ро
t.)	other additive before absorber	ou	ou	ou	on	ou	ou	on .	ou
Table 16 (cont.)	SO <sub>z</sub> added to	air	air	air	none	air	none	rie.	none
Table	Fuel	1. town gas	j. town . gas	k. town	1. town	m. town	n. town gas	o. town	p. town gas

2 tetrahydrothiophene

1 triethylene glycol

	conclusion	trace component is similar to t.h.t.	t.h.t. alone has little effect	
	visual effect	strongly luminous throughout flame	sl. yellow tip	luminous throughout flame
	other additive after absorber	t.h.t.	t.h.t.	t.h.t.
11.)	fuel through carbo- sillca sorb gel	yes	yes	yes
	fuel t	ou .	yes	yes
	other additive before	ou .	ou	ou
Table 16 (cont.)	303 added	air	none	air
Table	Fuel	• town gas	· CH4	· CH
:		<b>o</b> ' .	H	U)

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