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Education and Culture

Erasmus Mundus

Abstract

- CubeSats missions more and more demanding
- Current designs mostly limited to surface mounted solar cells
- ► Additional potential by deployable solar panels like in large satellites
- ► Further enhancements possible by proposed articulated solar panels
- Successfully validated prototype to vibrational loads during launch phase

Motivation

- ▶ Increasing power demands on small satellites
- ► Currently increase of solar cell area by deployable solar panels with fixed angle
- ▶ Proposed further improvements by adjusting solar panels for an optimized
- sun-incident angle

Subsystems

- ▶ Release Mechanism
- ▶ Required for releasing panel from locked configuration during launch
- ▶ Simple, light, small and reliable mechanism required
- Articulation Mechanism
- Proposed to increase the efficiency of solar cells
- ▶ Rotation of deployable solar panels in one or more degrees of freedom
- ▶ Simple, light and reliable mechanism required
- ► Type of Solar Cells
- ► Comparison of solar cells regarding costs, availability and achievable efficiency
- ► Control Mechanism
- ► To adjust the attitude of the solar panels to achieve optimal sun-incident angle
- ► Electronic controller embedded in on-board data handling systems preferable

Environmental Challenges

► Launch Environment

- ► Large vibrational load during rocket launch
- ▶ Risk of physical destructions by loosening of mechanical joints (e.g. screw connections)
- ▶ Risk of physical destructions by not improper locking of the solar panel and mechanical
- oscillations Orbital Environment
- ► Cyclic changes of temperatures during one orbit
- ▶ Induced stress on mechanical components by differences in thermal expansion
- ► Vacuum environment
- Outgasing of mechanical components
- Particle radiation Degradation of applied materials

Problem Statement

- ► Additional available energy: 12 Wh at AM0 by articulated solar panels
- ▶ Output voltage between 16 and 20 volts
- Embedded torque coil including interface
- ► Temperature sensor at back side of panel
- ▶ Possibility to print antenna circuit on or within the array substrate without degradation of array performance
- ► Electrical interface for antenna, torque coil, power and temperature sensor
- ► Size:

Design and validation of an articulated solar panel for CubeSats

Patrick Höhn

- \blacktriangleright 10x30x0.6 cm (placed completely outside of spacecraft) for a deployed solar array
- \blacktriangleright 10x30x1.6 cm for an articulated solar array
- ▶ Mass: 182.5 gram
- ► Compliance to the launch loads specified in NASA GEVS
- Compatibility with on-orbit temperatures
- ► Compatibility of thermal expansion / contraction of all materials used

Conceptual Design

- ▶ Hinge design with stepper motor enhanced by planetary gears for larger available torque and higher precision of sun-incident angle
- ► Attachment of solar panel by wire which is cut by heat winding
- ▶ Ultra Triple-Junction solar cells for highest efficiency
- ► Electronic controller for calculation of control signal to stepper motor
- ► CAD drawing of proposed mechanism:



► Constructed first prototype with mock-up solar panel:



Refined Specification

- \blacktriangleright Estimated Solar Cell Area: 0.058 m²
- ▶ Improvements by articulated Solar Panels:
- Fixed mounting parallel to +x panel: average of 286 W / m² per year
- Additional rotation around y axis: average of $817 \text{ W} / \text{m}^2$ per year
- Additional rotation around x axis: average of 496 W / m^2 per year
- Additional rotation around y and x axis: average of $835 \text{ W} / \text{m}^2$ per year



- Assembly of Solar Arrays: ► 8 solar cells
- ► Two solar panels
- ► Thermal Expansion:
- ▶ Maximum of 0.05 mm of mismatch due to differences in thermal expansion for a temperature difference of 161 K

Evaluation and Testing of the proposed Solution Classical numerical vibration analysis

- ► First step: sine survey (modal analysis), required to find oscillatory modes of analyzed system
- ▶ Right end of solar panel fixed

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► First ten calculated fundamental modes:

Mode number	Frequency
1	239.5 Hz
2	297.3 Hz
3	353.5 Hz
4	399.5 Hz
5	522.8 Hz
6	648.8 Hz
7	848.5 Hz
8	1041.9 Hz
9	1107.4 Hz
10	$1246.0 \; {\rm Hz}$

► First mode found in model analysis at approximately 240 Hz:



- ▶ Second step: classical analysis with constant excitation
- ► Resulting deflections:



- ► Analysis with randomly dynamically changing load to reflect launch conditions more realistically
- Power density function of oscillations as input
- ► Resulting deflections:



► Resulting mechanical stress:



- Numerical analysis of thermal expansion
- Temperature changes between 0 o C and 83 o C



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▶ Resulting stress in whole panel:



- ▶ Largest stress of 440 N / mm² within attachments \Rightarrow potential need to shift from Aluminum to
- Experimental vibration tests performed at Space Dynamics Lab of Utah State University
- ► Experimental setup with applied accelerometers:



Power Spectrum Density for random vibration test: Frequency Qualification ASD Level $(\mathbf{G}^2 / \mathbf{Hz})$ (Hz)20 - 50 + 6 dB / octave50 - 800 - 6 dB / octave800 - 2000 $14.1 \; {\rm G}_{rms}$ Overall ► First detected resonance frequency: 240 Hz

Conclusion and Future Work

- ► Large improvements by adding one degree of articulated degree of freedom
- Validated feasibility of proposed design
- ► Consistent results from numerical and experimental vibrational analysis
- ▶ Further tests for outgasing, particle radiation

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