## Evaluation of the Vertical Tube Evaporator and the Multistage Flash Desalination Processes

United States Department of the Interior



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### **FOREWORD**

This is one of a continuing series of reports designed to present accounts of progress in saline water conversion and the economics of its application. Such data are expected to contribute to the long-range development of economical processes applicable to low-cost demineralization of sea and other saline water.

Except for minor editing, the data herein are as contained in a report submitted by the contractor. The data and conclusions given in the report are essentially those of the contractor and are not necessarily endorsed by the Department of the Interior.

### COMPARISON OF CAPABILITIES AND COSTS

### OF THE

### VERTICAL TUBE EVAPORATOR (VTE) PROCESS AND THE

### MULTISTAGE FLASH DISTILLATION (MSF) PROCESS

### TABLE OF CONTENTS

		Page
1.0	Introduction -	1
2.0	Summary and Conclusions -	2
3.0	Technical Summary - 250 MGD Plants	
	3.1 VTE Plant Description - Process and Physical	8
	3.2 MSF Plant Description - Process and Physical	12
	3.3 Study Procedure	15
	3.4 Capital Cost Summary	25
	3.5 Product Water Cost Summary	26
4.0	Development Potentials - 250 MGD Plants	27
5.0	Bases for Cost Estimate - 250 MGD Plants	
	5.1 Capital Cost Bases	30
	5.2 Production Cost Bases	35
6.0	Site - 250 MGD Plants	40
7.0	Tables - 250 MGD Plants	
	7.1 Capital Cost Breakdown	41
	7.2 Product Water Cost Breakdown	42
	7.3 Operating and Labor Cost Estimate	44
8.0	250 MGD Conceptual Design - VTE and MSF Plants	
	8.1 Evaporator Structure	46
	8.2 Tube Inlet Nozzles	65
	8.3 Deacrators	67
	8.4 Pumps and Drivers	69
	8.5 Chemical Systems	72

TABLI	E OF CONTENTS (Cont'd.)	Dago
	8.6 Instrumentation	<u>Page</u> 75
	8.7 Electrical	79
	8.8 Material Selection	81
	o.o Faterial Berecolon	01
9.0	250 MGD VTE Plant Equipment List and Drawing Index	0.4
	9.1 Equipment List	86
	9.2 Drawing Index	94
10.0	250 MGD MSF Plant Equipment List and Drawing Index	
	10.1 Equipment List	106
	10.2 Drawing Index	114
11.0	Technical Summary - 2.5 MGD Plants	
	11.1 VTE Plant Description - Process and Physical	125
	11.2 MSF Plant Description - Process and Physical	133
	11.3 Study Procedure	`137
	11.4 Capital Cost Summary	140
	11.5 Product Water Cost Summary	141
12.0	Development Potentials - 2.5 MGD Plants	142
13.0	Bases for Cost Estimate - 2.5 MGD Plants	
	13.1 Capital Cost Bases	145
	13.2 Production Cost Bases	148
14.0	Site - 2.5 MGD Plants	151
15.0	Tables - 2.5 MGD Plants	
	15.1 Capital Cost Breakdown	152
	15.2 Product Water Cost Breakdown	153
	15.3 Annual Operating and Maintenance Labor Cost Estimate	155
	15.4 Cost Breakdown of Additions to VTE Plants	156
16.0	2.5 MGD Design - VTE and MSF Plants	
	16.1 Seawater Intake and Outlet	157
	16.2 Chemical Systems	158
	16.3 Electrical	160
	16 L Instrumentation	161

TABLE	OF CONTENTS (Cont'd.)	
	(00.00)	Page
17.0	2.5 MGD VTE Plant Equipment List and Drawing Index	
	17.1 Equipment List	165
	17.2 Drawing Index	173
18.0	2.5 MGD MSF Plant Equipment List and Drawing Index	
	18.1 Equipment List	182
	18.2 Drawing Index	185
19.0	Appendix	
	19.1 Charts	
	19.2 250 MGD VTE Computer Printout	
	19.3 250 MGD MSF Computer Printout	
	19.4 2.5 MGD MSF Computer Printout	

### 1.0 - INTRODUCTION

In July 1968, The Fluor Corporation, Ltd. was awarded a study contract from the Office of Saline Water (OSW) to determine, compare and evaluate the relative thermodynamic and engineering characteristics and economics of the Vertical Tube Evaporator (VTE) and the Multistage Flash (MSF) processes. The contract included a determination and assessment of the relative reliability, maintainability, and potentials of the two types of plants.

The evaluation is made on both large and small plants of each process. For the large plant comparison, OSW furnished conceptual designs of 250 million gallon per day (MGD) plants prepared by Oak Ridge National Laboratory (ORNL), in which the VTE plant used double-fluted vertical tubes. For the small plant comparison, Fluor was provided with A-E designs on three 2.5 MGD plants. Two of these were VTE plants furnished in two alternate designs made by the Stearns-Roger Corporation - one using smooth tubes, and the other using enhanced-surface tubes (double-fluted falling-film, and corrugated water-filled tubes). The MSF plant provided for comparison was the 2.5 MGD Universal Plant designed by Burns and Roe, Inc. for the OSW.

These designs were reviewed by Fluor and altered where necessary to make the process and engineering design bases the same for each plant. Capital costs and operating costs were then determined, using ground rules and procedures approved by OSW.

### 2.1 Large (250 MGD) Plants

### Summary

The large plant designs which were furnished for comparison are conceptual designs considered in some respects to be beyond the current limit of the state of the art; however, additional development programs should enable plants of this size to be built in the 1975-1980 period.

The reference Vertical Tube Evaporator (VTE) and Multi-Stage Flash (MSF) plants are each considered to be coupled to a nuclear-powered electrical power plant. Only the water plant portion of the complex is evaluated here, but steam and power costs and some capital costs are based on the dual-purpose nuclear power-water plant concept.

Each plant consists of four trains, with each train producing 62.5 MGD of fresh water. The performance ratio of each plant is essentially the same - 12.89 and 12.84 lbs. of product/1000 Btu for the VTE and the MSF plants, respectively.

Each train of the VTE plant combines a 50-stage flash feed preheater with a 15-effect vertical tube evaporator. The vertical effect tubes are copper-iron alloy with fluted surfaces to obtain high heat transfer performance; the horizontal feed preheater tubes are smooth 90-10 coppernickel. Each train of the plant is contained in a trapezoidal reinforced concrete structure, with the vertical tube effects located above the MSF preheater. The plant operates with a maximum brine temperature of 260 F.

Each train of the MSF plant consists of 48 heat recovery stages and 2 heat reject stages. Smooth tubes of 90-10 copper-nickel are specified. Each train is contained in a structure designed of steel for the portion of the plant operating above 200 F and of reinforced concrete for the low-temperature end.

Laboratory tests of enhanced-surface horizontal tubes indicate heat transfer rates much higher than obtained from smooth horizontal tubes. If field tests show that the high rates can be sustained, an MSF plant utilizing enhanced tubes could be substantially reduced in size and cost.

An obvious difference between the VTE and MSF plants is in the number of pumps needed for each. Principally because of its effect pumps, each train of the VTE plant requires 20 major pumps, compared to only 5 for each MSF train. These pumps are expected to be quite reliable, and unscheduled shutdowns should be infrequent. Nevertheless, the failure of an effect pump will stop operation of the entire VTE portion of the train involved; the failure of any other pump (in either plant) will completely shut down the train affected. Loss of the VTE portion of a train will cut the train production by about 83%, and of the entire plant by about 20%. To partially compensate for the greater risk of pump outage in the VTE plant, warehouse spares have been provided for the effect pumps, together with a special gantry for quick pump replacement. With this arrangement

### 2.1 large (250 MGD) Plants (Cont'd.)

downtime from failure of an effect pump should be limited to no more than 12 hours, including startup, and the 90% on-stream factor specified in the ground rules should be achieved. The average maintenance material cost of the more numerous pumps of the VTE plant is more than offset by the larger sizes of the MSF plant pumps and other components. However, the maintenance crew for the VTE plant is estimated to be larger because of the more numerous components to maintain.

The continuing escalation of prices must be considered in estimating the cost of building plants in the future. For the purposes of this study, the hypothetical plants are assumed to be started immediately. If such plants were to be built starting in January 1969, most material and labor expenditures for construction would occur in 1971, and water could be produced in 1972; therefore, costs have been escalated to those years.

Estimated costs, based on 4-1/4% interest for capital and a 30-year plant life, favor the VTE plant by a substantial margin. They are as follows:

	VTE Plant	MSF Plant
Capital cost (escalated to 1971), dollars	115,000,000	166,000,000
Product water cost (escalated to 1972), cents per 1000 gallons	25.9 to 25.0	30.2

Experience with the copper-iron vertical tubes of the VTE plant is limited and may not warrant the assumption of a 30-year life. For this reason, the water cost of the VTE plant has been estimated in two ways:  $25.9 \mbox{\rlap/e}/1000$  gallons if tubes must be replaced in 15 years, and  $25.0 \mbox{\rlap/e}/1000$  gallons if tubes last 30 years. The higher cost of water over a 30-year period reflects the estimated tube replacement cost that would be incurred during the 15th year. The replacement cost, in addition to new material and installation costs, includes the value of salvaged material and the cost of product lost during the replacement period.

The capital cost figures include extra contingency based on a "first-of-a-kind" plant of this capacity and conceptual design.

### Conclusions

For targe size plants, a VTE plant based on the reference design\* appears to have a distinct economic advantage over the corresponding MSF reference design, \*\* according to the findings of this study.

Use of enhanced surface tubes in the MSF plant would reduce its cost to some extent, but the cost of the VTE plant would still be substantially lower.

<sup>\*</sup>Design Study of 250 MGD VTE Desalination Plant by Oak Ridge National Laboratory, ORNL-4260.

<sup>\*\*</sup>Design Study of 250 MGD MSF Distillation Plant, ORNL-3912.

### 2.1 Large (250 MGD) Plants (Cont'd.)

Before large desalting plants of the reference designs can be considered ready for construction, however, additional research and development efforts are needed on the following:

### For the VTE Plant

- a. Life and heat transfer rates of the vertical enhanced tubes under actual field service conditions.
- b. An acceptable structural design that is safe in the event of a failure of an effect pump.
- c. Module tests for investigating problems in coupling an MSF section with the VTE section.

### For Both Plants

- a. Durability of reinforced concrete in a brine and in a distilled water and vapor environment when the reinforced concrete is in tension.
- b. The possibility of reduced cost through use of polymer concrete.
- c. Applicability of linings under development for steel and concrete suitable for 260 F temperatures.
- d. Cost of in-place reinforced concrete with a steel inner liner.

### 2.2 Small (2.5 MGD) Plants

### Summary

The three small plant designs which were furnished for comparison have been engineered and specified by others to the extent that guaranteed cost proposals could be provided. Two of the designs are for VTE plants - one with conventional smooth tubes and one with enhanced surface tubes - and the third is for an MSF plant. Each of the small plants consists of a single unit capable of producing 2.5 MGD of desalted water. The performance ratio (pounds of product water per 1000 Btu of heat input) of each plant is approximately the same - 11.6 for each of the two VTE plants and 11.5 for the MSF plant.

The two VTE plants were designed by and their cost estimated by Stearns-Roger Corporation. The MSF plant is a modified version of Plant No. 4 of the OSW "Universal Plant Design." This design was modified by The Fluor Corporation, Ltd., to correspond to the conditions of a hypothetical site, and to place the design on the same basis as the VTE plants. Fluor then made a definitive cost estimate of this "normalized" MSF plant for comparison with costs of the VTE plants. OSW agreed that the Stearns-Roger estimates of the selling prices of these plants would be an acceptable basis for comparison with Fluor's cost estimate of the MSF Universal plant. Fluor made a considerable number of process and physical changes to the Universal Plant Design No. 4 to place it on a basis comparable to the VTE plant. These changes are listed and discussed in Section 11.3, page 138.

Each of the small VTE plants has 14 effects and operates with a maximum brine temperature of 258 F. Unlike the large (250 MGD) VTE plant evaluated in this report, in which the feed preheating was done in an associated MSF unit, all feed heating in the small plants is done by means of the preheater tube bundles in each effect. All VTE tubes, whether smooth or enhanced, are of 90-10 copper-nickel, including the vertical falling-film tubes and vertical and horizontal exchanger tubes. For improved reliability, all pumps in the VTE plants are provided with installed spares. The 14 falling-film feed pumps have one spare for each two pumps; all other pumps in the plant each have an installed spare.

The small MSF plant consists of 39 flash stages for heat recovery and three for heat rejection. Maximum brine temperature is 250 F. The MSF plant includes five separate heat recovery vessels, a heat rejection vessel, and a separate brine heater. All heat-exchange tubes are smooth and made of 90-10 copper-nickel. Except for the brine recycle pumps, all pumps in the MSF plant are provided with installed spares. The brine recycle pumps are designed to operate in parallel, with each pump rated at 55% of the normal plant design requirement. Loss of one of these pumps would limit plant capacity to about 1 - 1.25 MGD.

### 2.2 Small (2.5 MGD) Plants (Cont'd.)

The relative reliability of the small VTE and MSF plants is judged to be approximately equal. With respect to reliability, there are 14 effect pumps in the VTE plants which must share a spare with another pump, while in the MSF plant one major service has no spare pump but does have two half-capacity pumps. All other components in the two types of plants appear to be comparable with respect to number and reliability of components.

With respect to maintenance, again the two types of plants appear to be about equal, the extra maintenance of the more numerous pumps and exchangers in the VTE plants being offset by the larger pumps and larger piping of the MSF plant. For installation in a remote location, however, the VTE smooth tube plant, and to a lesser extent, the VTE enhanced tube plant, does present maintenance difficulties because of its height. A mobile crane with a long boom is needed to remove effect covers, feed heater heads, and tube bundles. Purchase of this type of maintenance equipment usually cannot be justified for a small plant, and in some locations it cannot be hired on short notice.

The continuing escalation of prices has been considered in estimating capital costs and product water cost. For this study, it is assumed that if detailed design on a 2.5 MGD plant were started in 1969, most of the material and labor expenditures and the start of water production would occur during the following year; therefore, costs have been escalated to 1970.

The estimated costs, based on  $4\frac{1}{4}\%$  interest for capital and a 30-year plant life, are as follows:

	VTE Plant	VTE Plant	MSF Plant
	Smooth Tubes	Enhanced Tubes	Smooth Tubes
Capital costs - 1968	\$4,687,500	\$4,457,700	\$4,834,100
(escalated to 1970)	4,900,000	4,680,000	5,060,000
Product water costs per 1000 gallons - 1968 (escalated to 1970)	\$1.03 1.08	<b>\$1.</b> 02 1.07	<b>\$1.</b> 08

The capital costs and water costs developed in this study for 2.5 MGD plants are high in relation to recent bid prices for plants of this size; however, it must be remembered that these capital costs do include land and other owner costs plus all offsite facilities except the boiler plant, and escalation to 1970 prices. One major item of capital cost for all three plants in this study is a cost of more than half a million dollars for the submarine seawater intake line. Higher-than-usual land and site grading costs also contributed to high capital costs, and continuous chlorination added to the cost of production. For a Southern Florida or Caribbean site the capital costs of each plant would probably be about \$1,000,000 less than the 1968 capital costs shown above, and the corresponding water costs would be about 12¢/1000 gallons lower.

### 2.2 <u>Small (2.5 MGD) Plants</u> (Cont'd.)

### Conclusions

For plants of 2.5 MGD size, capital costs and production costs for the two types of plants evaluated (VTE and MSF) are very nearly the same. The enhanced tube VTE plant has the lowest cost, but the total spread in costs between the three plants evaluated is about 5%. This does not represent a significant economic advantage for either of the VTE plants over the comparable MSF plant.

One potential improvement which might modify the economic balance is the use of enhanced surface (corrugated) tubes in the MSF Universal Plant design. Several types have shown improved heat transfer characteristics in short-term tests. If extended tests bear out these advantages without disclosing serious drawbacks, enhanced tubes may net significant reductions in capital and production costs after allowing for higher pumping and tube costs.

Some discussion seems in order to explain why in very large plant sizes, the VTE process exhibits a striking advantage over the MSF, yet in small sizes, its advantage is marginal. There appears to be two main reasons for this. First, in the large plants, the percentage of the total capital cost represented by heat transfer tube bundles is very high, on the order of 40%. Therefore, any design change that results in an increase or decrease in heat transfer surface has a pronounced effect upon the total plant capital cost. In small plants, however, where the tube bundle cost totals only 20% of plant investment, plant cost is much less sensitive to this cost. Second, the 250 MGD VTE plant employs an MSF preheating scheme which improves the thermodynamic efficiency of the plant and, therefore, for a given performance ratio, reduces the total heat transfer requirement. The small VTE plant does not make use of this feature.

### 3.0 - TECHNICAL SUMMARY - 250 MGD PLANTS

### 5.1 VTE Plant Description - Process and Physical

### General\_Process

The process employed in this plant is a forward-feed, falling-film vertical-tube multiple-effect evaporator with a 50-stage flash evaporator feed heater. The vertical tubes are the "double-fluted" type to enhance heat transfer. The plant has a performance ratio of 12.89 pounds of product water per 1000 Btu thermal input. The process flow is shown on Dwg. V-3.

The incoming seawater is screened and chlorinated, then acidified and pumped to the deaerator where it constitutes approximately 70% of the feed to the evaporation cycle. Another portion of the incoming seawater flows through the tubes of the final heat reject condenser where it is warmed to 104°F. Two-thirds of this seawater is returned directly to the ocean as the major heat reject stream; the remaining one-third is acidified and pumped to the deaerator to provide the other 30% of the feed. The warm 104°F seawater stream is sprayed into the first side of the evacuated deaerator where it is deaerated and cooled to 77°F by the action of water vapor flashing from the surface of the droplets and carrying away dissolved gases. The colder 65°F seawater stream is sprayed into the second side of the deaerator where it is deaerated and heated to 77°F by the vapor which was evolved from the warm seawater stream on the first side of the deaerator. The noncondensibles are continuously removed from both sides by a three-stage ejector system with barometric condensers. Seawater at 65°F serves as coolant for the barometric condensers

The combined deaerated feed at 77°F is pumped into the condenser tube bundles of the multistage flash evaporator at Stage 50. The continuous tubing carries the seawater through all 50 MSF stages as its temperature is increased to 246.5°F by vapor that condenses on the tubes in each stage. The condenser tubing continues into the brine heater where steam from the power plant raises the brine temperature to 260°F. Steam condensate from the brine heater is returned to the power plant for boiler feed.

The brine leaving the brine heater passes through a pressure control valve and thence to a distribution chamber. One stream from this distribution chamber flows through an orifice into the first flash evaporator stage where steam flashes off at the reduced pressure, cooling the seawater approximately three degrees. The released vapor passes through the entrainment separator, condenses on the feed heater tube bundles, and falls into the product trough which is on the same floor level but separated from the flashing brine. The brine and condensate then flow separately through orifices into Stage 2 where both streams flash again. The vapor from both streams condenses on the feed heater tube bundles and falls down to join the condensate stream. This process continues through Stage 50 at which point the condensate, now at 90°F, is pumped to the product water system. The brine, now at a concentration 2.5 times that of normal seawater, and a temperature of 92.8°F, is returned to the ocean.

### 3.1 VTE Plant Description - Process and Physical (Cont'd.)

The 15 vertical-tube effects are located directly above the flash evaporator stages. The second stream of brine mentioned in the paragraph above flows from the distribution chamber to the Effect I pump and is then pumped to the brine chest above Effect I. Here it enters the vertical fluted tubes through individual nozzles. Steam supplied by the power plant to the steam chest of Effect I condenses on the outside of the tubes and is then returned to the power plant as boiler feed. The latent heat of condensation of the steam is transferred through the vertical tubes to the seawater inside, causing an equivalent amount of vapor to evaporate isothermally from the falling film of brine on the inside of the tube. Both steam and noncondensibles are carried down through the center of the tube. The vapor disengages from the brine in the space beneath the tube bundle, passes through the entrainment separator at the face of the Effect II bundle, and condenses on the vertical tubes in Effect II. The brine is then returned to the MSF brine stream. Noncondensibles are withdrawn at the outside top of each tube bundle. A similar process is repeated for each succeeding effect, with brine being withdrawn from the appropriate MSF stage, partially evaporated in the VTE effect, and then returned at the same temperature to the same MSF stage from which it was withdrawn. To prevent the slightly enriched brine from being again pumped to the same effect, it is returned by gravity flow through ducts in the stage floor at a point downstream of the suction connection of the effect pump. Except as described above for Effect I, condensed vapor from the outside surfaces of each vertical tube bundle flows by gravity to the product tray in the flash stage which is at the same pressure as the condensate. Vapor equalization passages are provided between each effect and its associated stage to accommodate design and operation uncertainties.

This process requires that there be a flash evaporator stage at the same saturation temperature as each vertical-tube effect. This means that the inter-effect temperature decrement must be spanned by some whole number of flash evaporator stages. Effects II through XII are each spanned by three stages; Effects XIII through XV each by four stages; and the first condenser by five stages.

### 3.1 VTE Plant Description - Process and Physical (Cont'd.)

### Physical Plant

The plant is housed in four trapezoidal reinforced-concrete structures, each containing one parallel and independent train. Each train has the multistage feed heater on the lower floor and the vertical tube effects on the upper floor, and is divided along the plant length into 50 flash evaporator stages and 15 vertical tube effects. The brine heater is integral and forms a continuation of the feed heater. The final condenser and deaerator are also contained within the evaporator shell.

The bundles of 10-ft-long, 3-in OD. double-fluted vertical tubes are constructed in 10-ft-deep by 20-ft-wide rectangular units; each unit has its own noncondensable gas cooler and vent removal section. Hook and vane entrainment separators are mounted at the face of the bundle. The vertical tube bundles will be prefabricated with appropriate tube spacing for each effect and will be installed with partial bundles as needed to provide the specified number of tubes for each effect. The width of the train (and of the entire plant structure) is determined by the number of bundles required to supply the total number of vertical tubes needed in each effect. Principal plant equipment is described in Section 9.1.

A standard vertical tube bundle is 10 ft deep (in direction of vapor flow) and 20 ft wide. The distance between the top and bottom surfaces of the tube sheets is 10 ft 3 in, and the tubes project 3/4 in below the bottom tube sheet and 1/4 in above the top tube sheet, as shown in Dwg. V-8. The polypropylene baffles separate each bundle into two regions - the main vapor-condensing section, and the noncondensable cooler and removal section. The unit bundle is of such a size as to be transportable, so that the machining and assembly operations can be done wherever is most economical. The vertical tube bundles contain an array of 3-in OD. by 0.049-in wall double-fluted tubes made of CDA 194 copper-iron alloy. Tube density varies from 13 to 8.72 tubes per square foot of tube sheet, progressing from Effects I to XV. Tube sheets are 1-1/2 in thick, the top sheet made of carbon steel and the bottom sheet of carbon steel clad on the brine side with 90-10 copper-nickel. The brine side of the top sheet is protected by the layer of cement in which the bases of the ceramic nozzles are imbedded.

Brine entry into each individual vertical tube is controlled by a ceramic nozzle cap mounted over it and held in place by a suitable cement. These nozzles perform the dual function of regulating the flow of brine to each tube and of distributing the brine over the inner tube surface. The top of each nozzle cap is shaped like a hat or slender dome, with a single 1/4-inch diameter hole formed on the side of the dome. The hole is directed slightly off-center so that the brine enters with a tangential component. The brine jet impinges on the inner surface of the nozzle cap and descends into the evaporator tube with a pronounced swirl, insuring complete wetting of the evaporating surfaces of the tube. For additional discussion on the ceramic nozzles, refer to Section 8.2.

### 3.1 VTE Plant Description - Process and Physical (Cont'd.)

The top tube sheet of each vertical tube bundle is made slightly wider than the bottom, which eases installation and provides for supporting the upper sheet on built-in shelves in the evaporator. Once the tube bundles are set in place, the support edges of the upper tube sheet are seal welded to the evaporator liner. Type 316 stainless steel flexible seals are welded between the tube sheets of adjacent bundles, both top and bottom, and between the bottom sheet edges and the evaporator shell.

The design contemplates that replacement bundles are fabricated and installed in the same fashion.

The feed heater bundles are continuous 371.6-ft-long tubes from stage 50 through the brine heater, and are installed by inserting the individual tubes through the stage separator sheets in place and rolling the tubes into the end sheets.

### Hook-and-Vane Entrainment Separators

The separators will be in standard sheets approximately eight stages deep by 9 ft long for the VTE effects and 6 ft long for the MSF stages. Sections approximately 10 ft wide will be assembled by hand using spacers and four tie rods, two at the bottom and two at the top of each unit. The units will be lowered into the partitioning walls and clamped at the top and bottom by two braces and bolts per 10-ft-wide unit. The units for the MSF stages will be mounted in a similar manner from the floor of the feed preheater section.

### Materials of Construction

The various materials of construction used in the VTE plant are described in Section 8.8 of this report.

### 3.2 MSF Plant Description - Process and Physical

### General Process

The process and mechanical design of this plant was developed by ORNL under a previous contract, and was reviewed and modified as necessary by Fluor for this study.

The process flow diagram is presented in Drawing M-3, and is of conventional MSF design. A 2500 MWt nuclear reactor generates high pressure steam which first drives a turbine generator and is then utilized to provide the thermal energy to the waterplant. ORNL has optimized the water plant for a performance ratio of 12.84 pounds of water per 1000 Btu thermal input.

The incoming seawater is first screened to remove ocean debris and chlorinated to control the growth of marine organisms within the plant. It is then pumped into the condenser tubes of a two-stage heat rejection section. In the reject condenser, the seawater receives a quantity of heat nearly equal to the plant steam input rate. Of this total flow, two-thirds is rejected to the sea. The remaining seawater, used for process feed, is acidified and pumped to the deaerator.

The deaerator is constructed as an integral part of the heat rejection section. It consists of two stages of brine spray with a countercurrent flow of water vapor, the latter being obtained from the final brine flash chamber. Condensation of most of the stripping vapor is accomplished by direct contact with treated, cold seawater. The inerts and remaining water vapor are withdrawn at a temperature of 80 F and discharged to the atmosphere by a three-stage ejector system.

The acidified, deaerated seawater feed is combined with a much larger stream of recycle brine and pumped at an average temperature of 91.6 F, through the tubes of the 48 heat recovery stages. Water vapor from flashing brine condenses on the tubes and heats the seawater inside the tubes to 238.7 F as the brine stream passes through the successively hotter MSF stages. From the condenser tubes, the seawater enters separate brine heaters and is further heated to 250 F with saturated exhaust steam from the power plant turbine generator. The hot brine then flows back through all the MSF stages, with additional flashing of water vapor occurring in each successively lower pressure stage.

The vapor from the flashing brine in each stage passes through demisters, condenses on the tubes, and is collected as product water in trays located beneath the tube bundles. This product is flashed from stage to stage in similar fashion to the brine, with the accumulated product pumped from the last stage to battery limits.

The plant is designed with the flashing brine flowing on two different levels. This is done to permit a narrower and therefore less costly evaporator building. Space is provided between sections of the upper brine tray to permit the passage of steam flashed from the lower level to the condenser above.

### 3.2 MSF Plant Description - Process and Physical (Cont'd.)

Interstage vapor pressure differentials provide the driving force to maintain the brine flow from stage to stage. Additional static head is provided by stage elevation changes at the low temperature end of the plant, where the stage-to-stage vapor pressure differential alone is insufficient to maintain brine flow. In the heat rejection section, the temperature decrement per stage is increased to promote brine flow.

The dissolved gases removed from the seawater in the deaerator are with-drawn from the plant by steam-jet ejectors. Traces of noncondensibles resulting from in-leakage and incomplete deaeration are carried with the water vapor to the condenser bundles. Gas cooling and gas removal sections in the interior of the condenser bundle serve to remove the noncondensibles which collect at each stage. The stages above atmospheric pressure are vented separately; the subatmospheric stages are served by the central ejector system.

### Physical Plant

The plant consists of four trains. Each train is 692 ft. long and is 90 ft. wide on the high-temperature end of the plant and 105 ft. wide at the low-temperature end.

The flash chamber design incorporates two flashing brine levels and a product water tray. The condenser tubing is 3/4" OD. x .035" wall, 90-10 copper-nickel alloy in the heat recovery section and 3/4" OD. x .049" wall, 90-10 copper nickel in the heat reject section, both with an assumed 30-year life. The flashing brine flow is ducted through the evaporator stages in two levels. The condenser tube bundles are located at the top of the evaporator structure, and the product water collected on the tube bundles drains into the product tray immediately below the bundles. The product is flashed to downstream stages in the same fashion as the brine.

Each plant train is stepped between the heat recovery and the heat reject sections so that condenser tubes can be inserted from either end of the heat recovery section and from deaerator end of the heat reject section. A single water box is used at the midplane of the heat recovery vessel.

The floor of the plant is a concrete slab with a 1/4" steel liner. The roof and the side walls of the high-temperature end (above 200 F) of the plant structure are of 1/2" arch construction steel. At the lower temperature end of the structure, 12" concrete with a 1/4" steel liner is used.

Separate brine heaters are provided for each train (two per train). The heater tubing is 3/4" OD. x .035" thk. 90-10 copper-nickel alloy. Each tube is 24.2 ft. long.

Principal plant parameters and items of equipment are described in the MSF Equipment List, Section 10.1.

### 3.2 MSF Plant Description - Process and Physical (Cont'd.)

### Tubes

The tubes for the evaporator condenser bundles are 90-10 copper-nickel alloy, 3/4" 0D, with .035" wall thickness in the heat recovery section and 0.049" wall thickness in the heat reject section. Continuous straight lengths of 238.5 ft. and 241.5 ft. will be utilized in the heat recovery section, and 67.5 ft. lengths will be used in the heat reject section.

The tubes for the brine heater are 90-10 cu-ni alloy, 3/4-in OD, with 0.035 in. wall thickness and 24.2 ft. length.

### Materials Selection

Materials used in the principal portions of the plant are discussed in Section 8.8.

### Structural Design

The evaporator structure is designed of structural steel for that portion of the plant operating above 200°F, and of reinforced castin-place concrete for the low-temperature end.

Steel stage walls are of stiffened flat plate supported from beams and columns.

Expansion joints are located at each tube sheet end to absorb the differential expansion between tubing and structure.

### Tube Bundle Design

Each of the four 62.5 MGD trains has five tube bundles in the heat recovery section and five tube bundles in the heat reject section, located above the product water trays. The continuous long-tube concept is used, with the heat recovery bundle being in two lengths. The first length is 238 ft 6 in long, extending from stage 1 through stage 25. The second length is 241 ft 6 in long, extending from stage 26 through stage 48. The heat reject tube bundles are 67 ft 6 in long.

Tube sheets at the water boxes are 3-in. thick steel, Cu-Ni clad on the brine side. Tube sheets at the stage walls are 1-in. thick steel. Intermediate tube supports are 1/2-in. thick steel.

The tube spacing for the 3/4-in. tubes is 1" at the high temp. end, 1-1/8" at the low temp. end, 1-1/16" in the heat reject section and .975"in the brine heater, center-to-center spacing in an equilateral triangular array.

The two tube bundle layouts for the heat recovery section have a conventional circular cross section and each contains 12,398 tubes. The tube bundle layout for the heat reject section has an elliptical cross section and contains 5,098 tubes.

### 3.0 - TECHNICAL SUMMARY - 250 MGD PLANTS

### 3.3 Study Procedure

### Process Optimization - VTE & MSF

The plant performance ratio has been established by ORNL at 12.89 pounds product water per 1000 Btu thermal input for the VTE plant and 12.84 for the MSF plant. These numbers are extremely close and for all practical purposes may be considered the same. It is our understanding that ORNL optimized the MSF plant using steam, utility, operating, and fixed costs which differ from those used in the current study. The VTE plant was then designed for the same performance ratio.

Ideally, each process should be optimized separately. In such case, the optimum plants could conceivably have widely different performance ratios. However, based on previous optimization work with sea water distillation plants, it is known that these plants display characteristically flat optimums. In other words, an independent variable may be changed by a significant amount with only an insignificant change in the optimized variable—water cost, in this instance. It is virtually inconceivable that a comparison made using equal plant performance ratios would be significantly different from a comparison between separately optimized plants.

A difficulty in comparing plants of different performance ratios and the same output is that differing unit heat costs have to be developed since the cost of thermal energy is a function of the size of a nuclear reactor.

In view of the preceding, Fluor believes that the equal plant performance ratio approach taken by ORNL is a valid one. To fully confirm this, we understand that the OSW intends to have ORNL perform additional optimization studies on both processes, using unit costs as developed by this study.

### Submergence Losses - VTE and MSF

Submergence allowances used in the MSF plant and the MSF section of the VTE plant are in accordance with the latest information on this subject developed by AMF under contract to the OSW. For the MSF section of the VTE plant, for example, submergence losses used by ORNL in the design, range from 0.8 to 1.9 F.

### Brine Distribution - VTE

Distribution of brine in pumping from the MSF stages to the VTE effects and in returning the brine to the MSF stages will have to be carefully designed in a real plant. Fluor believes the concept developed by ORNL is workable and that adequate distribution and mixing can be achieved. However, pressure drop calculations indicate the brine ducts carrying brine from each VTE effect back to the stage from which it was pumped are too small. Accordingly, a second duct the same size as the existing one was added on the opposite side of each train. It was necessary to locate certain horizontal portions of the second duct above the MSF floor to avoid interference with other transverse ducts.

Distribution connections from the transverse header back to the MSF stage must be sized for a higher velocity than exists in the header to assure adequate transverse distribution across the stage. Distribution of brine flowing from the MSF stage to the effect pump suction is probably less important, but would be achieved in the same manner.

The consequences of poor distribution have not been carefully evaluated, but incomplete mixing of the enriched returning brine with elevated boiling point would cause thermodynamic losses due to loss of thermal driving force. However, except for gross mal-distribution, the penalties involved are believed to be small.

### Stage Lengths - VTE and MSF

The MSF stage lengths shown on ORNL drawings do not agree exactly with those shown in the computer printouts. ORNL made small adjustments in these lengths to permit standardization of construction details and, in the case of the VTE plant, to facilitate vertical alignment of structural members in the MSF and VTE sections of the plant. Fluor considers this to be good practice. The effects on the thermodynamic efficiencies of the plants will be negligible.

### Recycle Pump Suction Conditions - MSF

The original ORNL design called for a stepped plant width when viewed in plan, with the train width narrowing suddenly at the point where the last stage joins the deaerator. At about this same point the brine drops several feet into the pump suction funnel, creating additional turbulence. This abrupt reduction in width and brine waterfall would create a violently disturbed flow and unsatisfactory pump suction conditions. The brine flow rate moving to the pump suction exceeds 2,500,000 lb/hr per foot of width. This flow rate is far beyond the highest demonstrated design (800,000 lb/hr per foot at the OSW's MSF Test Module), and is almost certain to result in unsatisfactory pump suction conditions. The maximum flow rate and other hydraulic features could only be determined by model test. Further, the indicated brine level of about 2 feet over a suction line diameter of 13.5 feet almost assures severe vortexing.

In order to improve the recycle pump suction conditions, Fluor has squared off the evaporator structure as viewed in plan, and contoured the evaporator floor to obtain a deeper brine pool and increased flow cross section leading into the recycle pump suction. The pump was also lowered several feet to provide a large, smoothly-contoured, low-velocity suction conduit designed to prevent vortexing and allow deentrainment of vapor. (See Dwgs. M-5 and M-7.) Placement of the suction line relative to the walls is also considered important but no changes were made. For a real plant design, model tests of the pump suction configuration would be required.

### Maximum Brine Temperature

The maximum brine temperature in the MSF plant is 250 F, whereas in the VTE plant it is 260 F. However, since the VTE is a "once-through" plant, the VTE brine heaters handle single-strength seawater, while the MSF brine heaters handle seawater concentrated by a factor of 1.72. For this reason, the VTE brine heaters actually operate further removed from CaSO4 hemihydrate scaling conditions than the MSF heaters. However, the VTE process may be susceptible to localized over-concentration in the boiling thin film, so that proper design would call for operation further away from scaling conditions. The Freeport VTE pilot plant has demonstrated long-term scale-free operation at temperatures of 260 F and higher, while the Point Loma MSF plant has demonstrated sustained 250 F operation. It is Fluor's opinion that the design maximum temperatures for these two processes are equitable, reflecting the level of current technology for the two types of plant.

To maintain product water costs on an equitable basis, the VTE plant is charged for its steam at a higher unit cost to reflect its higher level of available energy.

### Demisters - MSF

The "stepped" layout of the wire mesh demisters specified by ORNL for the MSF plant was checked to be certain that sufficient room was available to install the specified area of demisters. Layout sketches for Stages 48 and 50 were made; in both cases there was found to be ample room.

Fluor has calculated the pressure and temperature losses to be expected during steam flow through the wire mesh demisters and compared these figures with ORNL data. The Fluor approach to this calculation is to double the dry drop calculated from the Poppele and York (1) data.

The comparison between the Fluor and ORNL figures is tabulated below:

	Temperature	Loss, °F
<u>Stage</u>	Fluor	ORNL
1	.0184	.033
15	.047	.072
30	.110	.157
40	.224	•299
45	<b>.</b> 356	. 444
48	.416	.502
49	.400	•497
50	.408	.512

The agreement between these calculations methods is quite good, the maximum difference being about 0.1 F. Neither method is supported by actual operating data.

The ORNL calculated temperature loss is accepted.

<sup>(1)</sup> Poppele, E. W. and D. H. York, Chemical Engineering Progress, 59(6) 45-50 (1963).

### Bundle Loss-MSF

The pressure drop required for steam to penetrate through the condenser bundle to the annulus was calculated using a procedure developed previously by Fluor, and compared to the ORNL value. For Stage 48, the value predicted by the Fluor method was 0.55 F whereas ORNL calculated 0.51 F. This is considered excellent agreement; no changes were indicated in the ORNL design.

The above losses do not include the losses incurred in passing through the "air-cooling" section. This loss is usually greater than through the main field, but since the number of tubes in the air-cooling section is relatively few, the overall effect on reduction in MTD is not great.

In calculating mean temperature difference, ORNL employed the figures for total loss through the main field. An average or effective value is normally used. Since this is conservative and is similar to the procedure used on the VTE, no changes in the ORNL design were indicated.

### Evaporator Design Pressure - VTE and MSF

To determine the design pressures for the external walls for both plants, 5 F was added to the normal operating temperature, and the corresponding saturation pressure obtained from the steam tables and specified as design pressure. Since the brine typically has a BPE of about 2 F, the overall safety factor is about 7 F. For a real plant design, additional study would be required to determine whether this allowance is adequate to cover all dynamic conditions of startup, shutdown, and upset. For this study, neither the method nor the time was available. In a real plant, a process simulation computer program would probably be required.

### Design Pressure of Internal Walls - VTE and MSF

In an MSF plant, adjacent stages are connected by relatively large orifice openings. Although these openings are liquid-sealed during normal operation, in cases where differential pressure between adjacent stages rises, the seal will be lost and steam will blow through the orifice, thereby limiting the pressure differential that can exist. Standard practice in MSF plants, therefore, has been to design stage partition walls for the normal operating differential pressures with no factor of safety. This procedure has been employed in this study for the MSF plant and for the MSF section of the VTE plant.

In the VTE design, the design of the partition walls between effects is not so simple. Other means must be sought to prevent exceeding some design limitation on partition walls between effects during any or all excursions of process conditions. At least five possible ways of accomplishing this are listed in Section 8.1 "Evaporator Structure."

Evaluation of these schemes is clearly a complex problem beyond the scope of this study. At present, Fluor tends to favor oversized vapor equalizing ducts for the high temperature stages and using a design safety factor for the low temperature stages that would be safe under any conceivable condition. Rather than select any scheme, however, Fluor has included an arbitrary allowance of \$200,000 in the cost estimate to provide the necessary changes.

### Computer Printouts

The computer printout for the 250 MGD VTE plant used by Fluor in this comparative study program is different from the one included in the ORNL VTE Report 4260. The changes were necessary to put the VTE and MSF plants on an equal basis. After these process changes had been made, the altered conditions were transmitted to ORNL for incorporation into their computer program, and a new computer run was made. The changes included increasing the fouling factor from .0003 to .0005 in the preheater section, and increasing the nonequilibrium allowances to be consistant with those used in the MSF plant. The MSF computer output used for this study is unchanged from that shown in ORNL MSF Report 4214.

Both the updated VTE printout and the MSF printout are included in the Appendix of this report for ready reference.

### Heat Transfer Coefficient

The validity of the process comparison undertaken in the current study relies to a large extent upon the selection of proper and comparable overall heat transfer coefficients for each process.

In both processes, the fouling factor assumed by the investigator has a pronounced influence in the determination of overall coefficient. It is especially important in the VTE plant, since the film conductances are so high that fouling factor is a higher percentage of the total resistance to heat transfer than in the MSF.

In the recovery stages of the MSF plant and in the MSF section of the VTE plant a fouling factor of 0.0005 was assumed. In each MSF unit equal tubeside brine velocities were used. Overall coefficients computed on these bases are in general agreement with those in use by industry.

The ORNL method of determining overall heat transfer coefficient for the double-fluted VTE plant is presented in detail in their report, and will not be repeated here. In this procedure they evaluated data obtained in the General Electric pilot plant at Wrightsville Beach, North Carolina, and by ORNL in their single-tube test apparatus at Oak Ridge. The data evaluation did not permit separate identification of fouling factor. In addition, the reduction in thermal driving force due to pressure loss of steam through the inside of the tube was obscured and could not be identified. Further, inside and outside film coefficients were not separated. The correlation developed was simply for U value versus evaporation temperature, with approach temperature and salt concentration as parameters. Oak Ridge chose what appears to be a conservative interpretation of the data, with the U values they used falling below almost all of the data points.

Despite the conservative approach taken by ORNL, it is recognized that this is all laboratory test data, not long-term plant operating data. The principal uncertainty involved is whether the high transfer coefficients observed in the laboratories can be maintained over many years of operation, or whether a progressive accumulation of scale and/or dirt will reduce performance.

Long-term operation at Freeport has shown that U values for smooth tubes do not progressively deteriorate and that a constant heat transfer coefficient is soon reached. The double-fluted tube might be more sensitive to fouling for several reasons:

1. With the double-fluted tube, a small amount of fouling has a greater percentage increase in total resistance than with a smooth tube.

2. The grooves on the tube's inside surface might be preferential spots for scale or dirt to lodge, although no actual observations have been reported to support this conjecture.

There is now a double-fluted module in service at the OSW's Freeport facility. In the few months of operation so far accumulated, there has been no tendency for U value to deteriorate with time, according to the operators. Further, their calculations show they are achieving a U value of 1400 in Effect #9, which operates at a brine temperature of about 182 F. This agrees almost exactly with the ORNL correlation on which this design was based. Visual inspection of the inside of the double-fluted tubes after a few months operation showed them to be just as bright as the smooth tubes.

It would be desirable to develop a better fundamental understanding of the heat transfer mechanism involved with double-fluted tubes. However, this is a highly complex problem and a complete analytical treatment is not easily accomplished. For example, one phenomenon not readily understood is why higher heat transfer coefficients have been consistently observed for salt water than for fresh water.

In summary, the available evidence indicates that the overall heat transfer coefficients chosen by ORNL for the two plants are realistic and do not favor one process over the other. However, the inherent differences between the tube configurations and the processes themselves, add a factor of uncertainty to this comparison. Substantial improvement in comparison of the VTE and MSF processes must await reliable long-term data from an operating double-fluted VTE unit.

### Non-Equilibrium Allowances

The ORNL computer program employs a term called "alpha" which is a summation of chemical boiling point elevation, entrainment separator loss, and temperature loss from steam flowing across the condenser bundle. This figure is subtracted from the gross mean temperature difference (MTD) for each effect to determine the actual thermal driving force. The ORNL design was made prior to tests they performed on hook-and-vane type entrainment separators. These tests showed pressure drops through these devices to be substantially higher than ORNL initially believed. Consequently, it was necessary for Fluor to review the figures to see whether any design changes were indicated.

Chemical BPE data used by ORNL in these calculations are from the work of Stoughton and Lietzke at Oak Ridge. Fluor has no reason to question this data, and it is assumed to be reliable.

The ORNL tests and calculation procedure for pressure drop through the hook-and-vanes was reviewed and they are believed to be valid.

ORNL employs their own computer program to calculate the pressure drop experienced by steam passing through the condenser bundle. This program was not readily available to Fluor. Furthermore, the ORNL vertical tube bundle design specifies a varying tube pitch within each individual tube bundle, but from the available ORNL information Fluor was able to deduce only the average tube density within an effect. Accordingly, it was necessary for Fluor to assume a tube pitch for each section of tubing within an effect, using standard pitches specified by ORNL. With these assumptions, the pressure drops were computed, using a technique that had previously been developed by Fluor.

The bundle loss used by ORNL in their calculations is the total loss experienced by the steam in flowing all the way through the bundle. This is obviously conservative, since only the last tube row is exposed to the full loss, and every other row sees steam at a higher pressure. For the triangular-shaped tube bundle employed here, the effective loss for each section is approximately 60% of its total loss.

Fluor examined five different effects, and found the following results:

			EFFECT		
	II	VIII	XI	XIII	XV
Average tube density, tubes/ft. <sup>2</sup> (1)	14.6	13.4	12.3	11.1	9.35
S/D, 1st section (2)	1.15	1.3	1.3	1.3	1.45
S/D, 2nd section		1.15	1.15	1.15	1.30
$% = \frac{1}{2} \left( \frac{1}{2} \right)^{2}$ % of tubes in 1st section	100	16.4	49.9	90.6	70.7
% of tubes in 2nd section		83.6	50.1	9.4	29.3
Total temperature loss, °F	0.47	1.03	1.29	1.00	1.72
Effective temperature loss, °F	0.28	0.52	0.40	0.48	0.47

<sup>(1)</sup> Determined from computer printout, allowing 28 feet of plant width for interior walls and baffles.

Using the effective values for bundle loss instead of the total, a comparison of "alpha" values is made in the table below:

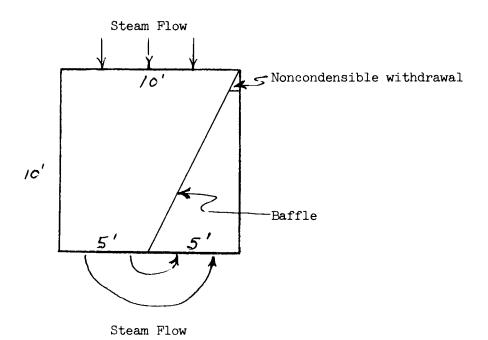
			EFFECT		
	II	VIII	XI	XIII	XV
BPE, °F	1.203	1.403	1.515	1.615	1.736
Hook-and-vane loss, °F	.009	.035	.065	.249	.878
Effective bundle loss, °F	.28	.52	.40	.48	47_
Total (Alpha)	1.492	1.958	1.980	2.344	3.084
"Alpha" used by ORNL	1.4958	2.3677	2.7476	3.0255	4.2511

<sup>(2)</sup> S/D is pitch-to-diameter ratio.

The ORNL values average significantly higher than those calculated by Fluor. Had the total temperature loss been used instead of the effective loss in the table above, the agreement between Fluor and ORNL would have been excellent. However, a portion of the difference would be required for the steam leaving the tubes of one effect to make three right angle turns before entering the next effect. Also, some loss is encountered in passing from the trapezoidal-shaped section of the bundle, reversing direction, then entering the triangular section. Even allowing several velocity heads for this, the numbers used by ORNL are concluded to be conservative. Consequently, no changes seem indicated in the ORNL design.

In calculating the bundle losses, it is apparent that ORNL used quite high values of Reynolds number for the steam flow. For example, using tube diameter as the significant length dimension, Fluor calculates for Effect II a Reynolds number of greater than 370,000 at the entrance to the tube field. This is higher than ordinarily used in MSF plants. No intrinsic reason appears why these high Reynolds numbers cannot be used. However, for a real plant design a suboptimization program is recommended to determine whether a wider tube spacing with reduced temperature loss might be more economical. It is interesting to note that Dr. A. Dukler of Houston University advocates the use of higher steam velocities with smooth tubes in order to increase film coefficients.

As advised by ORNL, the preferred tube layout can be represented by a number of 10-foot wide modules baffled as shown below:



Reverses Direction

### Evaporator Design

The MSF plant design calls for two separate brine levels. As far as is known, no plant of this type, either pilot or commercial, has ever been built. Consequently, this design, representing a departure from proven technology, was subject to review by Fluor. The incentive for using separate brine levels is evident—it permits the use of narrower and less costly evaporator structures by providing far more brine surface than could be obtained from a single level plant of equal width.

A satisfactory multilevel plant design must accomplish the following:

- 1. Brine from the brine heater must be divided so that each level receives its proper share of the total brine stream.
- 2. Brine tray design must be such that little or no brine splashes from the upper level to the lower level.
- 3. Upper trays must not unduly interfere with the flow of flashed vapor from the lower level to the tube bundles.

Because the final heat recovery stage is usually the limiting stage for the design, Fluor chose the design of stage 48 for review. The general arrangement of the brine trays is shown in Drawing M-8. The vertical velocity of steam passing between the two upper brine trays was found to be about 70 feet per second. To the best of Fluor's knowledge, this is considerably higher than is used in any commercial plant. Nevertheless, it is Fluor's opinion that if sufficient vertical distance or headroom is provided, deentrainment should be no worse than with a conventional design. Fluor believes that ORNL has provided sufficient height, but certainly a demonstration of this 2-level design would be required before a real plant could be built. At worst, it appears that several more feet of evaporator height would be needed to effect satisfactory deentrainment. This would, of course, result in an increase in shell cost, but the percentage increase would not be great and would have no more than a minor effect upon water cost.

Fluor believes that the brine tray design would be improved by extending the sides of the upper tray upward to join the sides of the product water tray above. This will prevent any splashing of brine from the upper tray to the lower tray, but will not interfere with the upward passage of steam.

Cross-troughs for equalizing liquid levels in the several brine trays were added by Fluor and included in this estimate.

# 3.0 - TECHNICAL SUMMARY - 250 MGD PLANTS

3.4 - CAPITAL COST SUMMARY

	VTE Plant	MSF Plant
Developed site cost:		
Land (for water plant) Site development Subtotal, developed site cost	\$ 1,780,000 8,358,100 \$ 10,138,100	2,600,000 9,488,000 # 12,088,000
Desalination plant cost (erected):		
Tube bundles Evaporator structure Other plant equipment	\$ 37,977,000 19,943,000 23,812,500	\$ 67,958,700* 32,710,700
Subtotal, desalination plant cost	<b>\$</b> 81,732,500	<b>\$122,968,8</b> 00
Engineering design, procurement, inspection, etc.	\$ 9,032,900	\$ 10,553,500
Interest during construction	\$ 5,989,800	\$ 8,672,000
GRAND TOTAL ERECTED COST (1968 DOLLARS)	\$106,893,300	\$154,282,300

\*Includes cost of separate brine heaters.

## 3.0 - TECHNICAL SUMMARY - 250 MGD PLANTS

## 3.5 - PRODUCT WATER COST SUMMARY

Ą	VTE Plant Assuming Vertical Tube Replacement in 15 Yrs.	VTE Plant Assuming No Tube Replacement	MSF Plant Assuming No Tube Replacement
Total Capital Cost - 1.968 Dollars 1971 Dollars <sup>(1)</sup>	106,895,300	106,893,300	154,282,300 166,000,000
Annual Cost, Dollars Capital charges(2)- includes interest, amortization, interim replacement and insurance	7,550,000	7,000,000	10,120,000
Annual charge to repay lor water production lost during tube replacement in 1983	115,500	1	;
Steam Electricity (3)	6,511,000 928,000	6,511,000 928,000	6,121,000
Materials and Supplies 77 Chemicals (4) Misc. Maintenance Supplies	2,678,500 820,000	2,678,500 820,000	3,193,000 820,000
Labor, including overnead(). Operation Maintenance	300,000 242,800	300,000 242,800	285,000 219,300
Total Annual Cost, Dollars	19,145,800	18,480,300	22,328,300
Annual Net Water Production, Gallons	81,950 × 10 <sup>6</sup>	81,950 × 10 <sup>6</sup>	82,025 x 10 <sup>6</sup>
Cents per 1000 gallons - 1968 dollars Cents per 1000 gallons - 1972 dollars(5)	23.4 25.9	22.6 25.0	27.2

<sup>2.3%</sup> escalation/yr

For fixed charges see Section 5.2 For breakdown see Section 7.2 Includes continuous chlorination of seawater 3000000

<sup>2.4%</sup> escalation/yr

### 4.0 - DEVELOPMENT POTENTIALS - 250 MGD PLANTS

### Use of Enhanced Tubes

In one respect, the comparison between the two processes is not on an equal footing. The VTE plant uses enhanced double-fluted tubes for two-thirds of the heat transfer surface, whereas the MSF plant employs all smooth tubes. Using 1968 technology, this is quite proper, since enhanced or extended surface heat transfer has been successfully demonstrated in the VTE process but not in the MSF. Because of inherent differences in the process, the double-fluted tube cannot be used in the MSF process. However, if some other type of enhanced tubes can be employed to advantage in the MSF process in the near future, it would improve the economics of the MSF process and perhaps influence the conclusions reached in the course of this study.

In this connection, Fluor has consulted with tube manufacturers regarding enhanced surface tubes suitable for horizontal orientation. These are being actively developed and show promise of substantial improvement in performance. The University of Michigan, in a recent publication<sup>1</sup>, reported on the results of an extensive test program which compared the performance of smooth tubes with a proprietary design of spirally-corrugated tubes<sup>2</sup> for use in seawater distillation and other heat transfer operations. ORNL also is known to be working on applying corrugated tubing to the MSF process.

The Michigan report includes data on inside and outside film coefficients and pressure drop for both plain and corrugated tubes. Based upon information presented in this report, Fluor has developed a design for a corrugated-tube MSF plant. Estimates of tube cost were furnished by the manufacturer.

The basis chosen for comparison was to design the corrugated-tube plant for a tubeside pressure drop approximately the same as in the smooth tube design, and to use the same fouling factor. This results in a lower superficial tubeside brine velocity; however, because of the corrugations, the turbulence and resulting resistance to deposition of dirt is probably no worse than for a smooth tube. Heat and material balance for the corrugated tube plant were taken as identical to the smooth tube design.

This comparison must be considered only approximate, since the design of the corrugated tube plant was prepared by designing Stage 25, comparing the stage lengths required, and factoring the stage lengths for all other stages. The salient design features of the corrugated-tube plant are as follows:

<sup>1</sup> E. H. Young, P. J. McParland, G. T. S. Chen, D. H. Young, "The Condensing of Steam on Horizontal Corrugated and Bare Tubes," University of Michigan, Ann Arbor, Michigan, Report No. 60, September, 1968.

<sup>2</sup> The Michigan tests were sponsored by Wolverine Tube Company, who developed this tube configuration and own the applicable patents - U.S. Patents 5,217,799 and 3,128,821.

### 4.0 Development Potentials (Cont'd.)

	Corrugated-Tube Plant	Smooth-Tube Plant
Tube material	90-10 Cu-Ni	90-10 Cu-Ni
Outside diameter	0.937" (1" nominal)	0.75"
Wall thickness	0.038"*	0.035"
Total tube length, recovery stages	327 feet	480 feet
Total tubes for 250 MGD, recovery stages	220,000	247,960
Differential head on recycle pump	207 feet	189 feet
Incremental KW for 250 MGD plant	4920	Base
Tubeside brine velocity (stage 25)	3.08 ft/sec	4.37 ft/sec

<sup>\*</sup>Tube is formed from 1" OD. .035" wall smooth tube.

Using an estimated tube cost provided by the manufacturer, and the comparative stage lengths calculated above as a basis, Fluor estimates the following savings would accrue if corrugated tubes were used throughout the MSF plant:

Evaporator shell	<b>\$</b> 6,400,000
Heat transfer tubing	8,200,000
Land and excavation	500,000
	\$15,100,000
Interest during construction	900,000
Total Estimated Savings	\$16,000,000

These figures include indirect costs, and assume that corrugated tubes are used in the brine heater as well as in heat recovery and heat rejection stages. This savings reduces the total estimated cost of a corrugated-tube MSF plant to approximately \$137 million, as compared to \$153 million for the smooth-tube plant.

Of course, the corrugated-tube concept, if it is advantageous for the MSF plant, could be applied to the MSF section of the VTE plant as well. However, since the preheater surface in the ORNL design is only about a third of the total surface, the cost improvement would be relatively less. Furthermore, because of geometric considerations it might prove difficult to adapt the ORNL evaporator configuration chosen for this design to an all-enhanced arrangement. If the MSF stages were shortened and the total length of the plant reduced in accordance, the plant would have to be widened in order to provide sufficient plan area for the VTE tubes. If the stage lengths were kept the same and

### 4.0 Decelopment Potentials (Cont'd.)

fewer preheater tubes used, velocity would be increased and pressure drop through the preheater section would be more than tripled. Of course, some basic changes in the evaporator configuration might be made to accommodate the use of corrugated-tubes in the preheater section, but consideration of such extensive changes is beyond the scope of this study.

This analysis shows the corrugated-tube design to good advantage. However, it must be recognized that the comparison is only approximate, and this tube has never been demonstrated. Further, the comparative pricing by the manufacturer, although presumably made in good faith, must be considered somewhat speculative at this time, as this tube is not in commercial production. Nevertheless, the tube appears to hold promise for flash evaporation plants and further investigation seems warranted.

#### 5.0 - BASES FOR COST ESTIMATES - 250 MGD PLANTS

## 5.1 Bases for Capital Cost Estimate - 250 MGD Plants

#### General Comments

The basic purpose of the estimates of the VTE and MSF 250 MGD plants is to compare the capital costs and the water costs for these two types of plants. The MSF design was more complete and therefore the MSF estimate is considered to be somewhat more accurate. However, both estimates were based on conceptual designs, and are subject to the limitations on accuracy implicit in such designs. In the following paragraphs specific portions of the capital cost estimate are discussed, with comments, where appropriate, on the assumptions and method of approach to each.

### Scope

The capital cost estimate for the 250 MGD desalination plant includes only the water plant and associated requirements. Although it is assumed the water plant is built in conjunction with a nuclear reactor and electric power generating plant, these are excluded from consideration in this study.

## Source for Process and Conceptual Designs

For the 250 MGD Vertical Tube Evaporator plant, the Oak Ridge National Laboratory Report 4260, dated August, 1968.

For the 250 MGD Multistage Flash Evaporator plant, the Oak Ridge National Laboratory Report 4214 - 1967 Technology.

#### Site Preparation

The large plants are assumed to be located on a Southern California coastal site, with a 200-ft. wide beach rising to an elevation of 35 ft. at the base of the coastal cliffs. At the cliff the terrain is assumed to rise 50 ft. to a level plateau at 85-ft. elevation, which extends inland beyond the site limits. The prepared site is to be at elevation 35 ft. An allowable soil bearing pressure of 4000 psf is assumed.

The major expense in site preparation is excavation and grading. For the VTE plant, more than 1 million cubic yards must be removed; for the MSF plant, because of its greater length, the figure is nearly 2 million cubic yards. It was assumed that this excavated material could be used either in other portions of this facility or on some location adjacent to the plant site. Some of the cut material could be used to build a temporary sea wall while the intake and outflow lines were being installed at the ocean breaker line.

Since there will be excavation for both plants which will be below sea level, sheet piling may have to be installed. Such areas include the pump pits and sumps, underground pipe lines and electrical ducts, and the intake structures. Dewatering points will be driven inside the piping and the water level pumped to below the work area.

# 5.1 Bases for Capital Cost Estimate - 250 MGD Plants (Cont'd.)

#### Structures

Concrete is priced on the basis that a batch plant will be set up at the jobsite, justified by the large amount of concrete required. Concrete unit costs will average \$147.00 a cubic yard for the MSF and \$114.00 a cubic yard for the VTE plant. The cause of the higher unit concrete costs for the MSF plant is that more reinforcing steel is required for adequate strength, and the construction of forms will be more difficult (for example, the product and brine trays). In contrast, the concrete pours are more massive for the VTE plant, providing lower costs per yard of concrete. In both plants, where concrete is lined by steel, it was assumed the steel shell would be put into place and braced to serve as forms for the concrete.

The cost allowed for evaporator lining material is on the basis of using polypropylene, which was specified in the reference design. This material may not be suitable for the plant operating temperatures, but it is assumed that a suitable material at no greater cost will be available before the plant is built.

The structural steel calculations are straightforward, with the exception of the roof structures. It appears that the connection for the roof structure could be simplified for less cost. Investigation into a bolted roof section is suggested.

#### Buildings

The only building directly considered in Fluor's estimates is a switch-gear building in the MSF case. This is for use of indoor type electrical equipment which in the voltages required is less costly than outdoor type. In this account Fluor has also included 1/3 of the cost for the buildings required for the entire complex, prorated to the water plant.

#### Piping

The piping estimate includes the lines from the seawater feed pumps, as well as the steel piping carrying flows to and from the pumps and evaporator structure. Also included is the cost of expansion joints associated with piping entering and leaving the structure.

#### Heat Exchanger Tubes and Tube Bundles

Both the horizontal and the vertical tubes for which estimates were secured were unusual items. The horizontal tubes were quoted as long tubes, extruded and coiled at the factory and shipped to the field in coil form. They would be straightened and inserted in the tube sheets in one operation at the site. The 3-inch vertical tubes were specified as being made of CDA alloy 194, with double-fluting to promote heat transfer.

## 5.1 Bases for Capital Cost Estimate - 250 MGD Plants (Cont'd.)

The copper-iron-phosphorous alloy CDA 194 is presently manufactured only by Olin Brass, a division of the Olin Mathison Chemical Corporation. Fluor has therefore relied solely on Olin for the prices of the 3-inch tubes. The fluting configuration was developed by General Electric; they have provided a quotation for fluting the tubes at the jobsite. Specifications of the smooth-walled tube priced for this study call for a minimum wall thickness of 0.065 inches (that is, the negative tolerance is zero). After the fluting operation, the developed wall thickness of the fluted tube will be 0.047 inches plus the positive tolerance of the original smooth tube. The fluting price allows for leaving the tube ends unfluted to facilitate rolling them into the tube sheets. Other vendors were contacted for the fluting operation, but none quoted prices lower than those from General Electric.

The preparation of the tube sheets accounts for a considerable portion of the VTE bundle expense. The estimate is based on 1-1/2" carbon steel A-285-C flange grade, with a bottom sheet having a 1/8" 90-10 copper-nickel cladding on the brine side.

Nine fabricators were contacted regarding the drilling of the tube sheets. None were found who felt they could drill 3-inch holes in a variable-pitch tube sheet with multiple spindles, using existing equipment. Most of the vendors felt they would do the work with several independent spindles working on one tube sheet, drilling and/or trepanning, with possibly some final boring and grooving. With nearly 1-1/4 million 3-inch holes to drill, there appears to be no basic reason why a multiple-spindle machine could not be designed to accomplish the purpose, especially if the tube pitch were held constant on any one tube sheet. For an optimized plant, an economic trade-off calculation should be performed, balancing the added cost of variable-pitch tube sheet drilling against the anticipated improvement in evaporator efficiency. Other hole-forming techniques were investigated, including casting and flame cutting plus reaming, but drilling or trepanning remains the method of choice.

The vertical tube bundle cost estimate is based on rolling the tubes into the tube sheets, rather than welding. The cost for hydrotesting is included. It is Fluor's opinion that the estimated cost for the VTE tube bundles is realistic, based on current technology, and that with more advanced drilling methods the cost of the tube bundles could be reduced.

#### Pumps

There was considerable variation in estimated costs of the very large pumps from the manufacturers contacted. The figures used in the cost estimate are average values.

## 5.1 Bases for Capital Cost Estimate - 250 MGD Plants (Cont'd.)

#### Instrumentation

The cost estimate for instrumentation is based on the contractor performing all field engineering work, including calibration, loop and function checks, the setting of pressure switches, alarms, shut-downs, and stroking and adjusting of solenoids and final control elements. It also includes an Instrument Engineer for consultation during start-up.

#### Common Facilities

The following facilities common to both the nuclear power plant and the water plant are included in the water plant costs at the percentage shown:

a.	Access railroad	- portion shown on plot plan
b.	Site preparation	- portion shown on plot plan
с.	Plant roads, parking & fencing	- portion shown on plot plan
d.	Compressed air system	<b>-</b> 33%
e.	Utility water system	- 33%
f.	Control Bldg., Administration Bldg.	
	and Warehouse	<b>-</b> 33%
g.	Sanitary sewer system	- 33%
h.	Building fire protection system	<b>-</b> 33 <b>%</b>

#### Owner's Costs

The following owner's costs are included in the plant capital costs:

- a. Cost of land occupied by Water Plant assumed at \$100,000 per acre.
- b. A&E costs, owner organizational costs, permits, etc.
- c. Bond financing costs
- d. Interest during construction
- e. Start-up costs labor, chemicals, etc.

#### Contingency

The allowance for contingency includes an additional risk-factor cost allowance for a first-of-a-kind facility.

## Product Water Treatment

The product water is assumed to be delivered at the plant boundary under a pressure of 25 psi. No capital costs are included in this estimate for product water treatment or storage.

## 5.1 Bases for Capital Cost Estimate- 250 MGD Plants (Cont'd.)

## Price Level

Labor and equipment pricing is based on 3rd quarter 1968 prices, then escalated to 1971 level based on a 3-year engineering-construction schedule. Annual escalation from 1968 to 1971 is assumed to be the same as the average annual escalation 1958 to 1968 in Southern California. Escalation is estimated to have averaged 4.2% annually for labor and 2.1% for material. It should be noted that escalation rates during the last five years are much higher than the ten-year average.

#### Labor Availability

Except for some shift work and overtime by concrete workers and riggers, this estimate assumes a 40-hour work week. Travel and subsistence to the site are included.

#### Equipment Pricing

Only U. S. manufacturers were contacted for equipment prices.

#### Spare Parts

An allowance for spare parts is not included except that in order to obtain the same reliability for the VTE plant as assumed for the MSF plant, warehouse spare pumps are included for the effect brine transfer in the VTE costs.

#### Taxes

It is assumed the owner will be tax exempt and, therefore, the California State Sales Tax of 5% on materials is not included.

### Form of Estimate

The breakdown is presented in essentially the form and detail shown in Appendix F of the 1965 Saline Water Conversion Report.

#### 5.0 - BASES FOR COST ESTIMATES - 250 MGD PLANTS

### 5.2 Bases for Product Water Cost - 250 MGD Plants

## Escalation

It is assumed that the plant could be operational in early 1972 if started in early 1969. Product water costs are estimated on the basis of the purchasing power of 1968 dollars, and operational costs then escalated to 1972, using the average annual escalation experienced from 1958 to 1968. Escalation is estimated at 4.2% annually for labor and 2.1% for material. It should be noted that escalation rates of the last five years have been much higher than for the ten-year period.

#### Cost of Money

Municipal bond financing is assumed. Bond interest rate is taken as 4.25%.

## Cost of Bond Financing and Interest During Construction

Underwriters and bond counsel fees and interest during construction are included in the capital costs.

### Repayment Period

Investment costs are amortized over a 30-year period corresponding to the estimated life of the desalination plants. The annual amortization factor of 1.71% is based on a sinking fund repayment method over a 30-year period at 4.25% interest.

## Interim Replacement

Both the VTE and the MSF plants are expected to have a useful lire of 30 years; however, a 0.35% factor is included for replacement of portions of the plants (other than tubes) not expected to last the full 30 years.

### Basis for VTE Tube Replacement Estimate

Copper-nickel tubes in both plants are assumed to last the life of the plant; however, the copper-iron vertical tubes of the VTE plant are assumed to require complete replacement after 15 years of service. To provide for this replacement, water costs of the VTE plant have been calculated on two different bases - first, assuming a 15-year life, and, second, assuming a 30-year life. To provide for replacement of the vertical tubes in 15 years, a replacement factor of 0.51% is included in the fixed charges in addition to the interim replacement factor of 0.35% for other portions of the plant. An annual fixed charge of 0.51% of the plant capital costs, when deposited at 4.25% interest, is estimated to provide sufficient funds at the end of 15 years to replace 65% of the tube bundles. Funds to replace the balance of the tube bundles, if borrowed at 4.25% interest, are repaid over the following 15 years by the annual fixed charge of 0.51% of plant capital cost.

# 5.2 Bases for Product Water Cost - 250 MGD Plants (Cont'd.)

The cost for replacing the VTE tube bundles after the 15th year has been estimated on a 1968 (3rd quarter) material and labor basis. The assumption is made that the four trains will require only one additional set of tube sheets, and that the removed tube sheets can be reused. By purchasing one additional set of tube sheets, the replacement tube bundles can be prepared prior to plant shutdown in order to minimize shutdown time. Once the tube replacement in the first train is complete, the tube sheets for that train will be cleaned and reused for the tube bundles for the subsequent train. No consideration has been given for the scrap material for the first set of tube sheets. It is assumed that 75% of the tube scrap will be recovered and sold at current scrap copper prices.

The tube bundle replacement procedure is expected to be carried out as follows. It is assumed that the roof can be removed in 40-foot sections and lifted to the edge of the building with a gantry crane designed for this purpose. It is further assumed that no lifting lugs are required for either the roofing section or the tube bundles. A crawler crane at the edge of the building will pick up the roofing section and load it on a waiting truck for transport to a temporary storage area. It is assumed that this can be done without seriously damaging either the roof liner or the outer insulation material. After the roof has been removed, the same gantry will be used to remove the tube bundles for transport to an adjacent work area.

The new tube bundles are now lifted into place, and the tube sheet seal welds and expansion joints completed. Sealing material is poured onto the top tube sheet, and new ceramic caps are placed over the tubes and embedded into the sealant-coated tube sheets. The roof is then replaced and the liner and insulation is repaired where necessary.

The cost estimate for the replacement is based on accomplishing this work on a 3-shift-per-day basis. Cost for purchasing, inspection, expediting, scheduling and overall project coordination is included in the replacement cost estimate.

# 5.2 Bases for Product Water Cost- 250 MGD Plants (Cont'd.)

## Fixed Charges on New Plant Costs

The total annual fixed charge is based on the following percentages of capital investment:

		VTE Plant 15-yr life on vertical tubes	VTE & MSF Plants 30-yr life on all tubes
a.	Cost of money	4.25%	4.25%
b.	Bond amortization factor (30-year basis)	1.71%	1.71%
c.	Plant replacement factor	None	None
d.	Plant interim replacement factor	•35 <b>%</b>	.35%
е.	Tube interim replacement factor	.51%	None
f.	Property damage insurance	.25%	.25%
g.	Taxes	<u>None</u> 7.07%	None 6.56%

## Energy Costs

Energy costs from the nuclear electric power plant were estimated by ORNL:

		VTE Plant	MSF Plant
Prime steam 342 F saturated, 121 psia Exhaust steam 270 F saturated 260 F saturated Electric power	¢/10 <sup>6</sup> Btu ¢/10 <sup>6</sup> Btu ¢/10 <sup>6</sup> Btu ¢/Kwhr	16.1 12.0 - 0.327	16.1 - 11.2 0. <b>3</b> 27
Chemical Costs (delivered to site)	Price, 100% B	asis	
Sulfuric acid (93%) Caustic soda Chlorine Anti-foam solution Sodium Sulphite Ferrous sulphate	\$/ton \$/ton \$/ton \$/1b \$/ton	24.00 73.00 72.00 .40 100.00 60.00	23.00 71.00 70.00 .40 100.00 60.00

### Maintenance Materials and Supplies

The annual cost of plant maintenance materials and supplies is estimated at 0.6% of direct capital.

## 5.2 Bases for Product Water Cost - 250 MGD Plants (Cont'd.)

#### Insurance

The annual insurance premium on the desalination plant is estimated to be 0.25% of capital cost.

#### Taxes

It is assumed that the owner will be a tax-exempt municipal or governmental agency, and that no city, county, state or federal tax or tax equivalent payments are applicable.

#### Load Factor

It is assumed the plants will be operating at full design load (250 MGD) all of the time that they are on stream.

### On-stream Factor

For a dual purpose (power and water) plant, it is assumed that the desalination plant will be available or on-stream 90% of the time. It is assumed to be shut down for maintenance, repairs, power plant outages, etc., the remaining 10% of the time.

In calculating VTE plant water costs, when a 15-year life is assumed for the vertical tubes, an additional penalty is incurred through loss of production revenue during the replacement period. This is estimated to be 2 months for each train. Loss of revenue is assumed to be the same as the water production cost. For convenience in calculating water costs the revenue which is lost in 1983 is converted to an equivalent 30-year annual cost.

#### Average Production Rate

Since the maximum daily gross capacity of each plant is 250 MGD, the yearly average capacity is 225 MGD (90% of 250). The net capacity is the gross capacity less plant useage for bearing cooling, glands, flushing, etc.

#### Product Water Pumping Requirements

Product water is delivered to the plant boundary at 25 psig and 90 F. No other pumping, passivation, or handling charges are included since conveyance pipeline and storage facilities are not considered plant costs.

# 5.2 Bases for Product Water Cost - 250 MGD Plants (Cont'd.)

#### Operation and Maintenance Labor

Operation and maintenance labor costs are based on a single staff and a single control room for both power and water plants. Personnel and estimated cost are shown in Section 7.2.

#### Owner's Administration and General Costs

These costs are estimated at 30% of operation and maintenance labor costs. The 30% includes payroll burdens of 20% (vacations, sick leave, State Disability Insurance, Unemployment Compensation Disability Benefits, and Federal Insurance Compensation Act.

## Interest on Working Capital

Interest on working capital is included in the Administration and General portion of operating costs, and is calculated at  $4\frac{1}{4}\%$ .

The location assumed for the 250 MGD desalination plants included in this study is at an unspecified location along the Southern California coast. The typical terrain in this area rises gradually from the water's edge to an elevation of 35 feet at the base of the coastal cliff, 200 feet inland. The cliff is 50 feet high, and the terrain is assumed to extend inland from the edge of the cliff at an elevation of approximately 85 feet above MLLW.

Transportation facilities are conveniently available to the site. U. S. Highway 101 and the Santa Fe railroad both parallel the coastline in this general area, and overland access to the site presents no problems. No dock or harbor facilities are presently available in this area to accommodate water transport. However, barging of large items could be accomplished at the expense of building barge-unloading facilities.

For the 250 MGD plants, the seawater pumps and screens will be located at approximately the original waterline, and the cold end of the evaporator structure itself will be 250 to 300 feet inland from the pumps. The original overburden will be excavated down to meet the design grade of approximately 35 feet above mean sea level, on which the distillation plant will be constructed. The typical sedimentary deposits in this area are readily excavated, yet provide good bearing capability.

With respect to the large (250 MGD) plants, this study is limited to consideration of the facilities required for the water plant, and excludes the nuclear power plant. The water plant facilities are shown on the plot plans (Drawings V-1 for the VTE plant and M-1 for the MSF plant). The MSF plant will require a plot area approximately 1,000 feet wide and 1,100 feet deep, measured from the water's edge. The VTE plant, being somewhat more compact, requires approximately the same 1,000 foot width but a depth of only 725 feet.

# 7.0 - TABLES - 250 MGD PLANTS

## 7.1 Capital Cost Breakdown

		VTE Plant Enhanced Tubes	MSF Plant Smooth Tubes
I.	Land (For water plant only)	\$1,780,000	\$2,600,000
II.	Site Development and Offsites		
	<ol> <li>Site excavation, roads, grading, parking, etc.</li> <li>Buildings (prorated)</li> <li>Submarine pipelines</li> <li>Seawater pump structure</li> <li>Screens and miscellaneous equipt. for intake         (seawater pumps below)</li> <li>Utilities (service water, service air, sewers, etc.)</li> </ol>	527,000 156,800 5,416,800 1,569,100 604,300 84,100	891,600 165,000 6,206,700 1,532,400 593,300 99,000
	Subtotal - Section II	<b>\$</b> . 8 <b>,</b> 358 <b>,</b> 100	<b>\$</b> .9,488,000
III.	Conversion Plant (Erected on site):		
	<ol> <li>Evaporators         <ul> <li>a. Bundles</li> <li>b. Structure (incl. shell)</li> </ul> </li> <li>Brine heaters (For VTE, incl. in Evaporator cost)</li> <li>Noncondensables removal equipment         <ul> <li>a. Condensers</li> <li>b. Ejectors</li> <li>c. Sump pumps</li> </ul> </li> </ol>	37,977,000 19,943,000  111,400 192,300 81,800	63,487,700 32,710,700 4,471,000 97,000 277,500 80,400
	d. Miscellaneous equipment  4. Pumps and drivers a. Seawater intake b. Booster c. Brine recycle d. Make-up brine e. Deaerator feed f. Blowdown g. Effect h. Product water i. Miscellaneous	1,040,000 253,500  1,172,600  235,300 3,960,600 580,200 25,700	47,700 1,105,000  4,121,900  546,300 339,000  580,200 25,200
	5. Make-up Pretreatment Systems a. Chemical feed system (incl. pumps and tanks) b. Deacrator internals 6. Excavation, foundations and concrete (excl. structure) 7. Piping and pipe supports a. Semmeter and brine b. Product water c. Steam and condensate d. Noncondensables	773,300 1,330,500 210,200 3,016,000 354,000 354,300 698,500	783,800 1,618,400 290,700 2,842,400 213,900 669,500 1,145,700
	8. Electrical installation (incl. controllers and switchgear) 9. Instrumentation a. Instruments and controls b. Control panel c. Instrument air system (incl. compress., receiv., driers) 10. Insulation 11. Painting Subtotal - Section III	3,601,900  2,559,300  85,800  173,600  2,672,000  329,700  \$81,732,500	2,995,300  1,677,200 64,300 88,000 2,247,300 442,700  \$122,968,800
gav v r •	Prelimering, Design, Procurement, Construction Management and Prespection, Owner Organization Costs and Owner Startup Costs	\$ 9,032,900	<b>\$</b> 10 <b>,553,5</b> 00
v.	Interest Examp Construction	<b>\$</b> 5,989,800	\$ 8,672,000
	GRAND TOTAL - 1968 DOLLARS	<b>\$</b> 106 <b>,</b> 893 <b>,</b> 300	\$154,282,300

# 7.2 Water Cost Breakdown - 250 MGD Plants

# 1968 Production Costs

		VTE Plant	MSF Plant
Daily Production On Stream Factor Gross Production Prod. Water Usage in Net Production	MGD % MGY Plant MGY MGY	250 90 82,200 250 81,950	250 90 82,200 175 82,025
Steam - Saturated Heating Steam	lbs/hr x 10 <sup>6</sup> Btu/hr x 10 <sup>6</sup> ¢/Btu x 10 <sup>6</sup> \$/hr \$/day \$/yr	7.165 6690 12.0 803 19,270 \$6,330,000	7.124 6715 11.2 752 18,050 \$5,940,000
Ejector Steam  Subtotal - Steam	lbs/hr x 10 <sup>6</sup> Btu/hr x 10 <sup>6</sup> ¢/Btu x 10 <sup>6</sup> \$/hr \$/day \$/yr \$/yr ¢/1000 gals	0.12 143 16.1 23 552 \$ 181,000 \$6,511,000 7.96	0.12 143 16.1 23 552 \$ 181,000 \$6,121,000
Power	kw ¢/kwh \$/hr \$/day	36,000 0.327 118 2,820	61,000 0.327 199 4,790
Subtotal - Power	\$/yr ¢/1000 gals	\$ 928,000	\$1,570,000 1.92
Subtotal - Energy	\$/yr ¢/1000 gals	<b>\$7,</b> 439,000 9.09	\$7,691,000 9.38

# 7.2 Water Cost Breakdown - 250 MGD Plants (Cont'd.)

# 1968 Production Costs

Chemicals Acid - 66° Be (93%)	tons/day \$/ton \$/day \$/yr	VTE Plant 225 24 5,400 \$1,780,000	MSF Plant 270 23 6,200 \$2,040,000
Caustic - Dry Basis	tons/day \$/ton \$/day \$/yr	8 73 584 \$192,000	10 71 710 \$233,000
Chlorine	tons/day \$/ton \$/day \$/yr	16 72 1150 \$378,000	22 70 1540 \$506,000
Miscellaneous Chemicals Anti-foam Sodium sulfite Ferrous sulfate Subtotal - Chemicals	\$/day \$/yr \$/yr	1000 \$ 328,500 \$2,678,500	1260 <u>\$ 414,000</u> \$3,193,000
Miscellaneous Maintenand Materials Charts, Rags, Gaskets	#/1000 gals	3.27	3.89
Lube Oil, Paint, Fitte & other supplies Subtotal - Misc. Mate	) \$/day	2500 \$ 820,000 1.00	2500 \$ 820,000 1.00
Operating & Maintenance Operation Maintenance Supervision Subtotal - O&M Labor	Labor	187,750 158,250 71,800 417,800	178,250 138,250 <u>71,800</u> 388,300
Grand Total, Water	\$/yr \$/1000 gals \$/yr \$/1000 gals	125,000 \$ 542,800 .66 \$11,480,300 14.0	116,000 \$ 504,300 .61 \$12,208,300 14.9

<sup>1</sup> Excluding fixed charges on capital cost

## 7.0 - TABLES - 250 MGD PLANTS

7.3.1 ANNUAL OPERATING AND MAINTENANCE LABOR COST ESTIMATE FOR 250 MGD  $\underline{\text{VTE}}$  PLANT (1)

	3060	Personnel	Personnel	
	1968	Req'd for Pwr. & Water	Chargeable to	Annual Cost
	Annual Rate So. Calif.	Plant	Water Plant	Water Plant
Dlont Monogement	So. Calli.	FIGURE	Water France	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,
Plant Management				
Station Supervisor	\$20,000	ļ	1/2	\$10,000
Chief Engineer	15,000	1	1/2	7,500
Secretary-Typist	6,600	1	1/2	3,300
Clerk	6,000	1	1/2	3,000
Guard	7,000	_5_	2-1/2	17,500
Sub-total		9	4-1/2	41,300
Technical Staff				
Nuclear Engineer	15,000	1	None	
Water Plant Engineer	15,000	1	1	15,000
Health Physics Superviso		1	_ None	27,000
Chemist	9,000	1	1/2	4,500
Laboratory Technician	8,000	2	1 ′	8,000
Clerk	6,000	<u>1</u>	1/2_	_ 3,000
Sub-total		7	3	30,500
Operating Staff				
Shift Supervisor	12,500	5	2 <b>-</b> 1/2	31,250
Water Plant Operator	10,000	9	9	90,000
Power Plant Operator	10,000	9	None	70,000
Auxiliary Operator	9,500	5	2-1/2	23,750
Instrument & Monitoring			,	• , , ,
Technician	9,500	9_	4-1/2	42,750
Sub-total		37	18-1/2	187,750
Maintenance Staff				
Maintenance Supervisor	12,500	2	1	12,500
Mechanic-Machinist	10,000	7	3-1/2	35,000
Pipefitter-Welder	10,000	5	2-1/2	25,000
Electrical Mechanic	10,000	5	2 <b>-</b> 1/2	25,000
Instrument Mechanic	10,000	5	2 <b>-</b> 1/2	25,000
Helper	8,500	3	1-1/2	12,750
Storekeeper	9,000	2	1	9,000
Laborer	8,000	2	1	8,000
Janitor	6,000	2	_1	6,000
Sub-total		33	16-1/2	158,250
Total Direct Labor		86	42-1/2	417,800

<sup>(1)</sup> When operated in conjunction with a 2500 Mwt Nuclear Power Plant

7.3.2 ANNUAL OPERATING AND MAINTENANCE LABOR COST ESTIMATE FOR 250 MgD MSF PLANT(1)

		Personnel		
	1968	Req'd for	Personnel	
	Annual Rate	Pwr. & Water	Chargeable to	Annual Cost
	So. Calif.	Plant	Water Plant	Water Plant
<del></del>	00. 00111.	1 1 4410	Water Traine	Wat OCT Train
Plant Management				
Station Supervisor	\$20,000	1	1/2	\$10,000
Chief Engineer	15,000	1	1/2	7,500
Secretary-Typist	6,600	1	1/2	3,300
Clerk	6,000	1	1/2	3,000
Guard	7,000	5	2-1/2	
	1,000			17,500
Sub-total		9	4-1/2	41,300
Technical Staff				
Nuclear Engineer	15,000	1	None	
Water Plant Engineer	15,000	1	1	15,000
Health Physics Superviso		1	None	17,000
Chemist	9,000	1		l. 500
Laboratory Technician			1/2	4,500
Clerk	8,000	2	1	8,000
Clerk	6,000	1	1/2_	3,000
Sub-total		7	3	30,500
Operating Staff				
Shift Supervisor	12,500	5	2 <b>-</b> 1/2	31,250
Water Plant Operator	10,000	9	9	
Power Plant Operator	10,000	9	None	90,000
Auxiliary Operator	9,500	9 5		00 750
Instrument & Monitoring	9,500	2	2 <b>-</b> 1/2	23,750
	0.500	_	/-	
Technician	9,500	<u>7</u>	<u> 3<b>-</b>1/2</u>	33,250
Sub-total		35	17-1/2	178,250
Maintenance Staff				
Maintenance Supervisor	12,500	2	1	10 500
Mechanic-Machinist	10,000	2	1	12,500
Pipefitter-Welder		5	2 <b>-</b> 1/2	25,000
	10,000	6	3 2	30,000
Electrical Mechanic	10,000	4		20,000
Instrument Mechanic	10,000	3	1 <b>-</b> 1/2	15,000
Helper	8,500	3 2	1-1/2	12,750
Storekeeper	9,000	2	1	9,000
Laborer	8,000	2	1	8,000
Janitor	6,000	2	1	6,000
Sub-total		29	14-1/2	138,250
Total Direct Labor		80	39-1/2	388,300
			,	5 ,5

<sup>(1)</sup> When operated in conjunction with a 2500 Mwt Nuclear Power Plant

## 8.1 Evaporator Structure

### Applicable Codes and Standards

Uniform Building Code

ACI Standard Building Code Requirements for Reinforced Concrete (ACI 318-63)

AISC Specification for the Design, Fabrication and Erection of Structural Steel for Buildings

ASME - Used for stability of arched steel roof only

Note: Since the conditions inherent in the vessel design which require lower stresses do not exist in this structure, the full stresses recommended for a given material in the above codes have been used.

#### Materials

Design is based on use of the following materials:

- a. Structural steel, including liner plate and arched roof Carbon steel, ASTM Grade A-36
- b. Reinforcement steel

ASTM A-15, intermediate grade

c. Concrete

Compressive strength (F'c) of 4000 psi

With limestone aggregates and calcareous sands and gravel to improve heat resistance of concrete.

With cements low in tricalcium aluminate content, such as Type V Portland cement, or Type II Portland cement as second choice. This will reduce concrete damage from sulfates present in the brine.

With pozzolan addition and mixed with a low water-cement ratio. Concrete should be dense and well-cured to resist water permeation.

With all reinforcement covered with a minimum of 2 inches of concrete; 3 inches is recommended.

#### Reinforced Concrete Design Comments

There has been evidence reported from Bureau of Reclamation tests 1 that reinforced concrete specimens showed only limited amounts of deterioration when submerged in hot flowing synthetic brine. Nevertheless, Fluor still has reservations about the use of reinforced concrete as a highly-stressed construction material under actual desalination plant operating conditions.

In a plant such as the 250 MGD VTE plant, reinforced concrete is simultaneously exposed to elevated temperatures, varying mechanical stresses, differential fluid pressures across the member, and an environment of hot flowing seawater. In the Bureau of Reclamation tests the specimens were exposed to elevated temperatures, no mechanical stresses, no differential fluid pressures across the specimen, and an environment of hot flowing synthetic brine. In addition, the test specimens were cast under controlled laboratory conditions, whereas the actual plant members would be subject to the less rigorous quality control of field construction. There are enough differences between these two sets of conditions to raise doubts as to the applicability of the reported findings.

It has been reported by some writers that concrete performs a corrosion-inhibiting function which acts to prevent the corrosion of its reinforcing steel. In the Bureau of Reclamation tests it is reported what several reinforced concrete specimens were deliberately cracked before being exposed to brine corrosion tests. The presumption is that these cracks allowed the brine to penetrate to the reinforcing steel, with the objective of observing their corrosion rates, and comparing them with those of uncracked specimens. As far as can be determined, however, the results of the corrosion tests of cracked specimens have not been reported.

Aside from the problem of possible corrosion of reinforcing steel, there is also the question of the ability of the concrete itself to withstand the hot brine environment. After conducting tests at various temperatures, the Bureau of Reclamation reported:

- a. After 18 months at 290 F in recirculating brine, concrete specimens exhibited deterioration to a depth of approximately 1/4 inch.
- b. After 6 months at 203 F, 225 F, and 250 F, in recirculating brine, specimens exhibited a degree of surface change. Because of the short test period, the Bureau said it was not possible to predict the useful life of concrete under these test conditions.

In view of the uncertainties with respect to the resistance of reinforced concrete structures to a hot seawater environment, Fluor feels that further testing will be required before building a plant of this size designed for a 30-year life.

<sup>1</sup> Evaluation of Concrete and Related Materials for Desalination Plants, General Report No. 37A, U. S. Department of Interior, Bureau of Reclamation, June, 1968

### Soil - Assumed Conditions

Allowable bearing pressure 4000 psf

Allowable passive (lateral) resistance 200 psf/ft depth.

Coefficient of friction 0.4

### Loadings

Roof (live load) 20 psf
Wind 15 psf

Earthquake (no allowable stress increase

used) 0.2 g

Pressures See Figs. 8.1.1 and 8.1.2

Temperature expansion coefficient

for both steel and concrete

0.65 x 10<sup>-5</sup>

#### Loading Combinations - Both Plants

D + L D - dead load

D + L + E L - live load

D + L + T E - earthquake

 $D + L + P_O + E$  T - temperature

 $D + L + P_O + E + T$   $P_O$  - design operating pressure

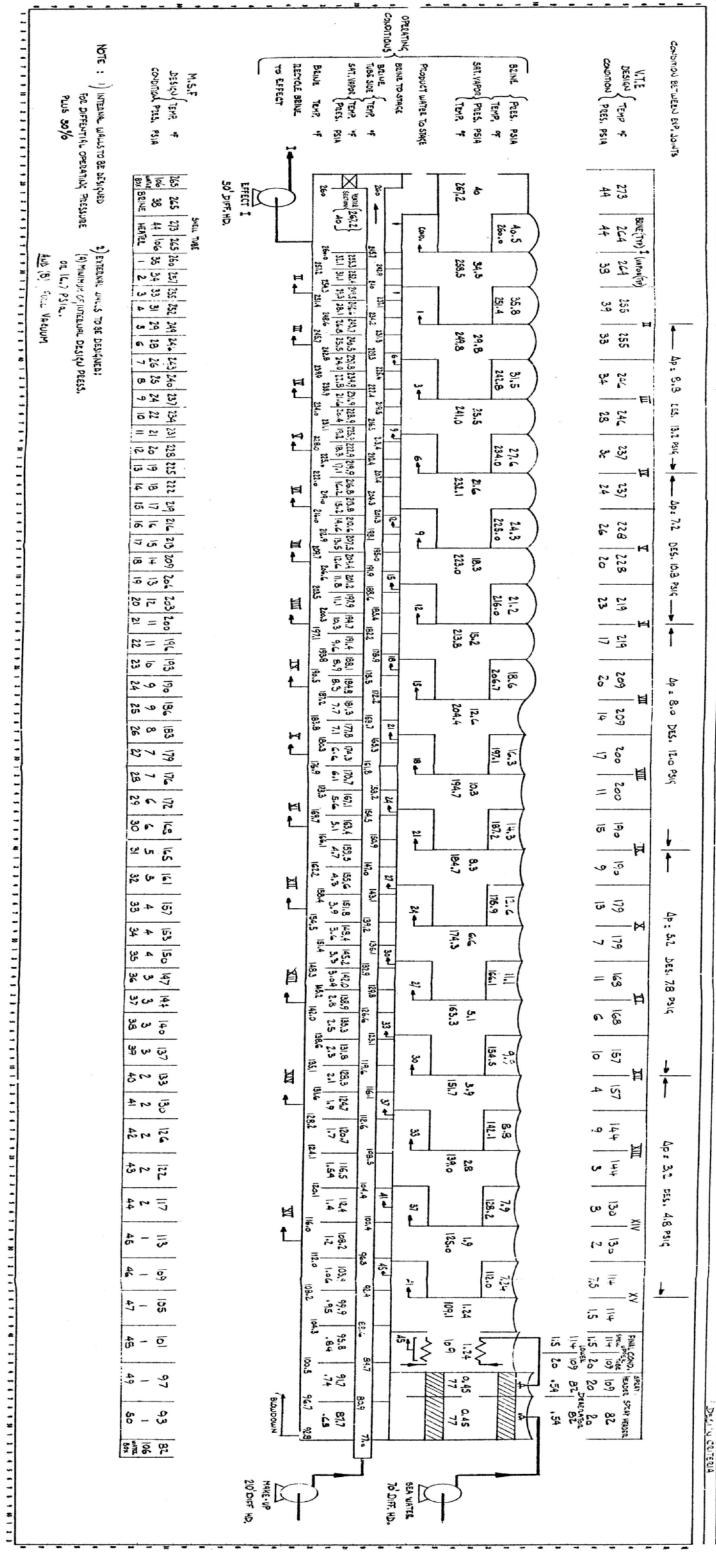
 $D + P_{f.v.} + T$   $P_{f.v.} - full vacuum$ 

#### VTE Design Pressure

Determination of design pressures to be used for the VTE structure proved to be a complex problem. For this study, the structure was designed using the design criteria shown in Figure 8.1.1. The internal walls were designed for the normal differential in operating pressure plus a 50% safety factor.

It is recognized, however, that under certain upset conditions this might not be sufficient. Several means to provide complete structural safety which were considered are as follows:

a. Design the internal walls to resist the maximum differential pressure that would exist during any credible incident. To establish these conditions would require a careful and detailed study of the process dynamics for quite a complicated system (VTE plus MSF) for many different operating situations, including failure of any one or combination of items of mechanical or electrical equipment, instrumentation, power, or steam. Costs to strengthen the walls in the higher temperature effects for such contingencies would be substantial because of the large pressure differentials.

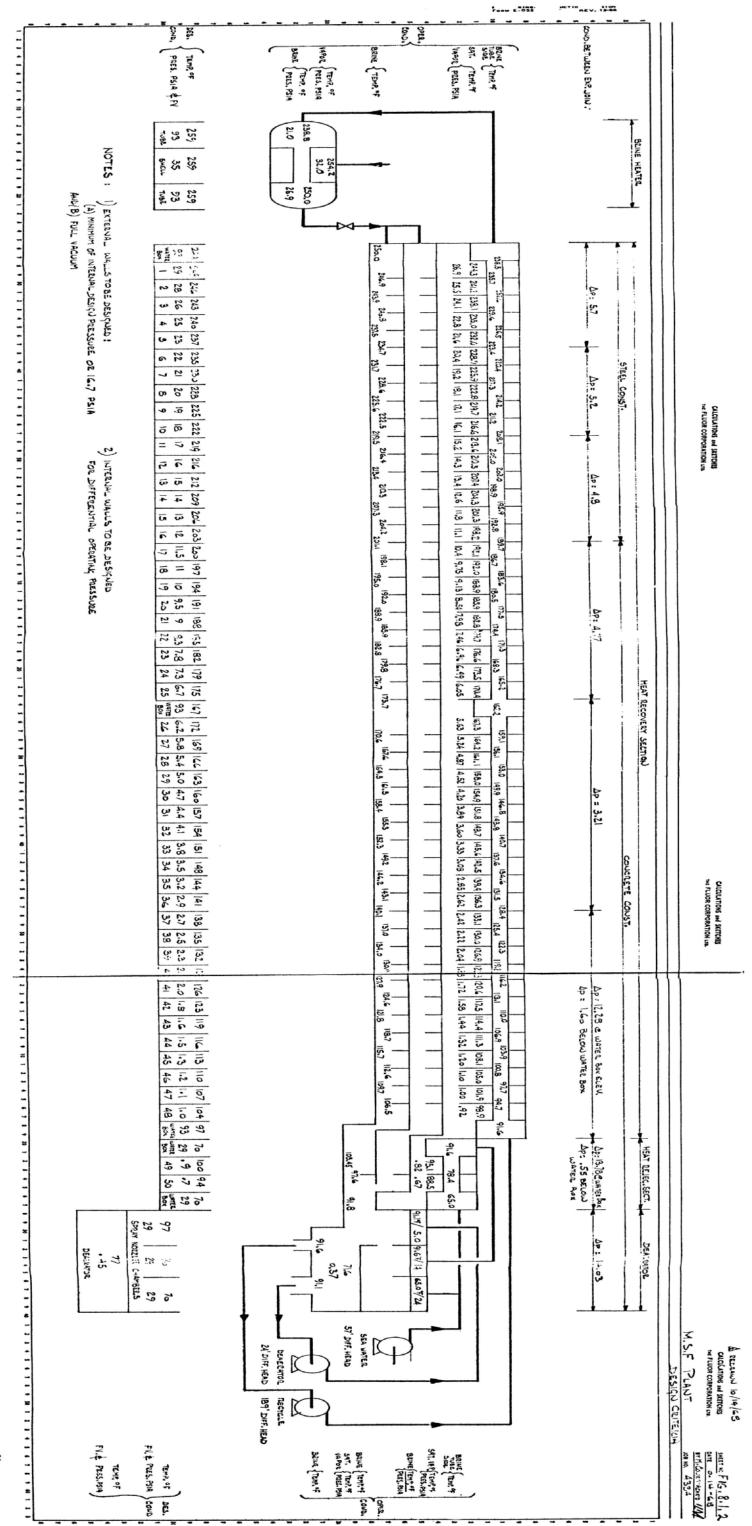


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- b. Provide internal relief valves in the partition walls set to open at design differential pressure plus some modest safety factor such as 50% of the normal differential. Inspection and/or adjustment of these would be impossible during operation. Furthermore, unless these were to reseat themselves, a complete plant shutdown would be necessary every time an overpressure incident occurred. The cost in lost production of one such incident would probably justify a more expensive system not requiring shutdown. In Fluor's view rupture devices are, therefore, an unsatisfactory answer. Standard spring-loaded relief valves are also impractical due to the extremely high volumes of steam that would have to be relieved to limit the threatened over-pressure.
- c. Oversize the brine ducts which carry brine from each effect to the MSF stage normally operating at the same pressure, plus including a safety factor of, say 50% in the design of the effect partitions. In this manner any tendency for overpressure in an effect will be relieved by conveying the excess steam to the corresponding MSF stage. Part of the excess steam delivered to the stage will be relieved by increased condensation in the MSF stage and increased brine flow to the next stage. If stage pressure exceeds the brine leg in the orifice, the water seal will blow out and the steam will pass through to the next MSF stage. This scheme is complicated by the difficulty of predicting two-phase (brine plus steam) flow through the brine duct, as well as by the uncertainties of undesirable dynamic effects, such as water hammer and brine pendulation.
- d. Oversize the vapor equalizing ducts between each effect and its equal-pressure stage counterpart, allowing for a reasonable overpressure in each effect, such as 50% of the normal differential pressure. In this manner, excess steam in, say, Effect (n) would be transmitted to the corresponding flash stage, through the next several flash stages to the flash stage connected to Effect (n+1), and thence to Effect (n+1), where it would act to decrease the differential pressure across the partition separating Effects (n) and (n+1). The vapor equalizing duct sizes provided by ORNL would have to be increased by a factor of 10 or 20 times to serve this purpose.

The most practical routing for such large ducts appears to be in the floor between the VTE and MSF portions of the plant. A double slab floor would be required, with the ducts located between the slabs. The MSF portion of the plant has a total headroom of 10'-6", which seems to be more than required. Accordingly, sufficient vertical distance appears to be available for such ducts. However, because of the longitudinal displacement of many of the effects with respect to their corresponding stages, at some positions in the length of the evaporator three separate parallel ducts would be required in each train. It might become a problem to find sufficient room for all the ducts,

particularly at the low temperature end of the plant where the specific volumes of the steam are high and the differential pressures are small. Although a complete analysis has not been made, it is possible that installation of these ducts might allow the plant to continue in operation even though an effect pump was out of service.

Provide water or brine seals between adjacent effects, arranged to blow out the water seal at a differential pressure between effects of, say, 150% of the normal differential pressure. Partition walls between effects would be designed to withstand the differential pressures permitted by the water seals. In this scheme, the downstream leg of the water seal carries a higher level than the upstream leg, and would be designed for a "no-flow" condition. Such a method could better be applied to the low temperature end of the plant where differential pressures are only a few feet. For the higher temperature effects, the design becomes more difficult because the higher differential pressures require greater brine depths in the water seal. Another problem associated with this design is that when relieving, the leg must pass both liquid and steam, so that possible undesirable dynamic effects must be considered. Another problem is the fact that the liquid in the top of the downstream leg is exposed to the pressure in the downstream effect and so will boil and pass some steam into this effect Consequently, additional brine will flow into the leg to replace the material evaporated. The density of the downstream leg will be reduced since the fluid will be two-phase, and a greater free-board will be needed to prevent carry-over. Thermodynamic losses will be incurred due to flashing of brine directly from one effect to the next rather than through the flash train.

Evaluation of each of these schemes is beyond the scope of this study. For cost estimating purposes, an allowance sufficient for a combination of scheme <u>a</u> and <u>d</u> is included in the cost of the structure.

External walls of the VTE plant were designed for an internal pressure equal to normal operating pressure (but not less than 2.0 psig), and also for full vacuum.

Effect chambers over the vertical tubes, deaerator chambers, and water boxes were designed for pressures equal to 120% of the normal discharge pressure of the pump feeding them. The 20% overpressure is to accommodate the pump shut-in head in case it should be inadvertently applied to the system.

#### MSF Design Pressure

Design pressures in the MSF plant were more readily determined than in the VTE. Since adjacent stages are connected by orifices normally closed with a liquid seal, any interstage pressure that was substantially different than normal would be able to relieve itself by blowing out the water seal and relieving into the adjacent stage. Internal walls were therefore designed for normal operating differential pressures. External walls were designed for an internal pressure equal to operating pressure, but not less than 2.0 psig, and also for full vacuum.

Deaerator chambers and water boxes were designed for pressures equal to 120% of the normal operating head of the pump supplying them. The added 20% is to care for the possibility of pump shut-in head being inadvertently applied to the system.

#### Problems Encountered in Design Check of VTE Structure

The VTE evaporator structure, like the MSF, is essentially a large pressure vessel, with added problems created by its higher temperatures and pressures. Even more important, however, is the fact that the differential pressures between effects in the VTE structure are much greater than the differential pressures between stages in the MSF plant. The high temperatures to which the structure are exposed require several expansion joints. The segments created by these transverse expansion joints tend to be pushed apart where the internal pressure is positive, or forced together by atmospheric pressure in the case of internal vacuum. These pressures generate horizontal forces which in some cases are nearly five times as large as those generated by design magnitude earthquakes.

The above conditions created problems in the VTE structure that, in Fluor's opinion, were not **ad**equately covered by the ORNL design. These include:

to resist them, all the horizontal forces generated by pressure and earthquake loadings must be balanced by the lateral passive resistance of soil against the keys beneath the structure, plus the friction between the structure and the supporting soil, to prevent the segments from moving. In the present design, there is not enough resistance under the structure segments (in cold and hot ends) to prevent sliding.

#### Example:

Consider the portion of the evaporator structure lying between expansion joints and including Effects II, III and IV, and a typical bay 10' wide. This segment has a length of 44' and a height of 33'. The loads on the 10' bay are:

Total weight = 1057 Kips per 10' width

= 597<sup>k</sup> (from 2.64 ksf net pressure) Total horizontal force H

+ 134k (from earthquake)

= 731 Kips

Frictional resistance of soil =  $.4 \times 1057 = 423^{K}$ 

Passive resistance of soil

 $= 3 \times 60 = 180^{k}$   $= 603^{k}$ bearing against ducts

Total resisting force

Since the horizontal force of 731k is greater than resisting force of  $603^{k}$ , the segment will slide.

- Overturning Large horizontal forces also tend to overturn the segments. To prevent overturning, a heavy slab is added (8' thick at the hot end). This overturning, however, increases soil bearing pressures in some segments to double the permissible ORNL's plan to use piles to reduce local high bearing pressures is acceptable, although in most cases mixing piles with spread footing is undesirable because of the likelihood of differential settling. For a real plant a thorough soil investigation and foundation analysis will be required before this procedure could be accepted.
- c. Beams The horizontal forces mentioned above must be transmitted from the effect walls to the foundations through a system of columns and beams. This system is inadequate and cannot be economically modified to become adequate.

Example:

Consider a column in Effect II:

Weight of structure, etc. = 146 Kips Internal pressure = 33 psia Atmos. pressure = 14.7 psia

Net pressure = 18.3 psi or 2.64 ksf outward

Total pull (uplift) on column =  $2.64 \times 10 \times 11 = 280^{k}$ Net uplift = 280-146 = 134 Kips

Horizontal force/column = 731 Kips (see item a)/seg.= 731/4 = 183 Kips/col.

Then the minimum moment in column =  $183 \times 10/2 = 915^{1}$ . To resist an axial tensile force of 134 Kips and a bending moment of 915 ft-Kips would require a column so massive as to be impractical.

- d. <u>Differential expansion</u> A relatively thin stage wall heated on both sides will expand quite rapidly during plant start-up while the thick slab supporting it, and heated on only one side, will not. Differential expansion will cause very high stresses in the walls, expecially at the tray openings, causing large cracks or even local failure of the wall (it will shear at the base).
- e. <u>Common division wall</u> Using a common division wall between two trains without an expansion joint is impractical.
- f. Underground ducts which project a key of concrete into the soil will tend to prevent movement of the floor slab due to passive or lateral resistance of soil. This will either reduce the effectiveness of the expansion joints or will shear the soil in front of the ducts and they will lose their effectiveness as shear keys.
- g. <u>Lining of floor slab</u> with 1/4" steel plate creates one of two problems:
  - (1) If the plate terminates at columns and walls, there will be a break in the plate's continuity at the junction of columns and walls with the floor slab.
  - (2) If the plate is continuous, the connection of columns and walls to the slab is complicated.

## Changes to the VTE Structure Introduced by Fluor

Because of the number of major structural problems which appear in the ORNL design, it is Fluor's opinion that fundamental design changes would be advisable before a plant were to be built, rather than simply reinforcing and strengthening the present design. However, such a major redesign is beyond the scope and available time for the present study. Therefore, modification to the ORNL design has been elected so as to provide a cost estimate within the required time frame. These are based on what are considered to be reasonable assumptions, and checked with short-cut calculations, but have not been thoroughly analyzed. The cost estimate is based on this approach and Fluor believes the estimated costs are sufficient to build a VTE plant following a design that Fluor would sponsor. The modifications are as follows:

- a. To prevent sliding, shear keys were added. No safety factor was included in calculating frictional resistance against the soil, although a safety factor of 1.5 on coefficient of friction is usually recommended.
- b. To provide for excessively high soil bearing pressures, the concept of combining piles with spread footings was assumed workable, and additional piles were introduced.
- c. To resist the large horizontal forces, the beam and column system was replaced in the VTE portion with shear walls, and diagonal struts were added in MSF portion. The large outward pressures causing tension in walls and columns necessitated the use of heavy tie rods in the walls and steel pipes in the columns.
- d. No provision was made in the cost estimate for means of controlling the differential expansion between walls and floor slab. It is assumed that suitable design modifications could solve this problem without additional cost.
- e. Two independent walls were used between the trains to accommodate lateral and differential longitudinal expansion.
- f. No provision was made in the estimate for dealing with problems created by ducts.
- g. A continuous steel floor plate was assumed.

#### Recommendations for Design of the VTE Structure

Several recommendations are introduced as major design changes which Fluor believes would provide a more workable VTE structure and at less cost. A more complete evaluation of these concepts would be required before Fluor could include them as sponsored designs. Fluor recommends the evaporator floor under the MSF portion of the VTE plant be built of double-slab construction, with two slabs of approximately equal thickness.

A shear key midway between expansion joints anchors the upper slab to the lower, with provision made for sliding between slabs at other points. The steel plate liner is installed between the two slabs instead of on top of the evaporator floor. Figure 8.1.3 illustrates this modification.

The use of a double-slab floor for the VTE structure shows the following advantages:

a. Expansion joints are a troublesome and inherently weak portion of the evaporator structure. Because the length of segments between transverse expansion joints can be increased by this construction, the number of joints can be reduced to two, or possibly even one. At the same time, the net differential pressures at expansion joints are effectively reduced.

#### Example:

Consider the existing expansion joint in Effect II:

Internal pressure = 33 psia

External (atmospheric) pressure = 14.7 psia

Resultant differential pressure = 18.3 psi, or 2.65 Ksf

Then consider the proposed expansion joint in Effect VI:

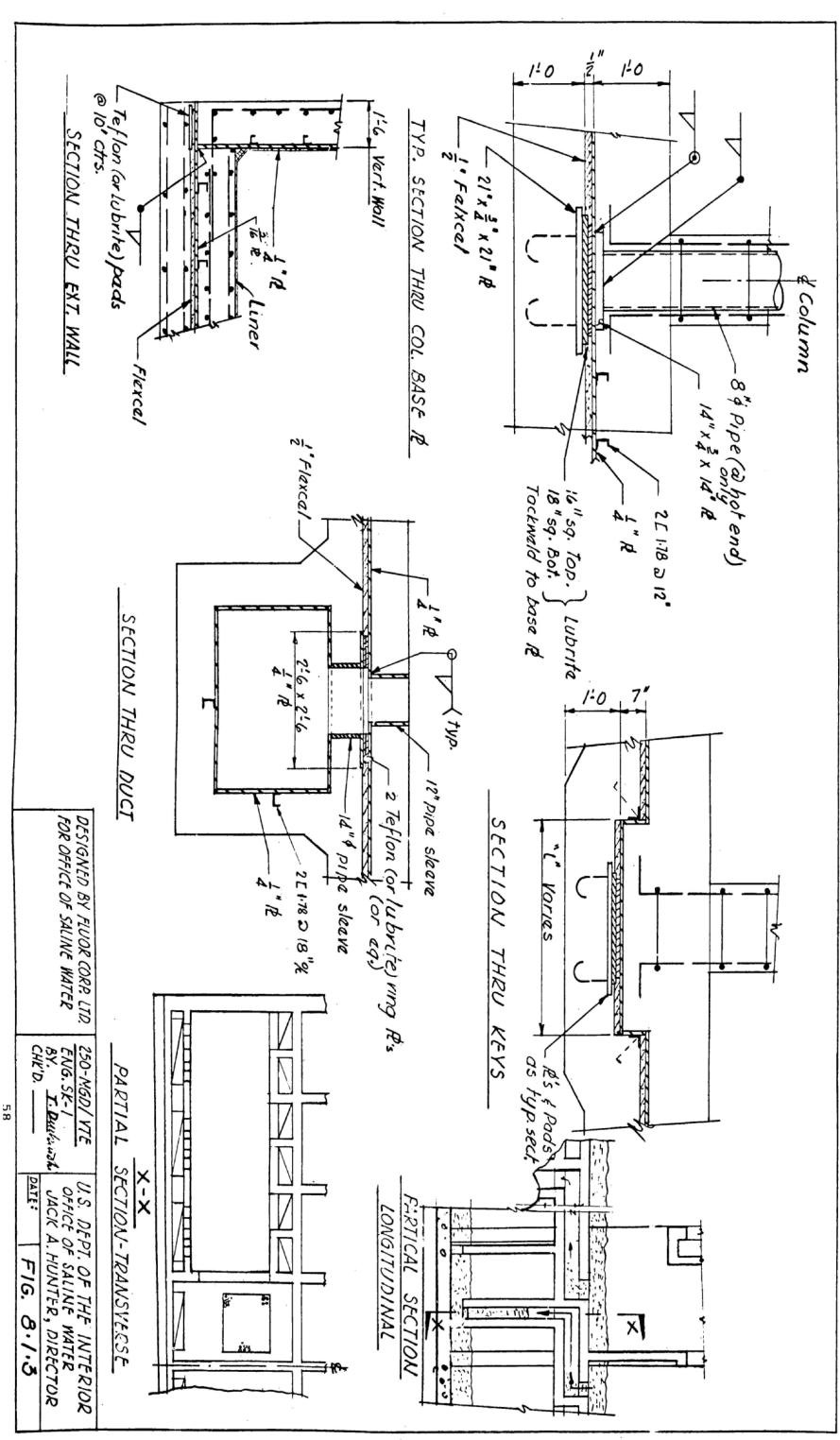
Internal pressure = 17 psia

External (atmospheric) pressure = 14.7 psia

Resultant differential pressure = 2.3 psi, or 0.33 Ksf

Thus the net pressure differential providing the horizontal forces in the second case is 1/8 that of the first case, while the evaporator segment against which the force is acting is three times as heavy. Therefore, overturning and sliding problems are correspondingly reduced.

- b. The structure will be held together by the bottom slab, eliminating the risk of sliding or overturning.
- c. It will be possible to separate the brine return ducts from the main structure.
- d. It will make the expansion of the top slab more nearly comparable to that in the upper structure, thereby greatly reducing internal stresses around the joints.



- e. The reduction of net horizontal forces will make it easy to transfer horizontal shears through external walls and one additional longitudinal wall, eliminating costly and cumbersome diagonals.
- f. The 1/4" steel plate will be used more effectively. Installation between the slabs eliminates the problem of sealing continuity around walls and columns.

Fluor recommends that the VTE plant brine return ducts be removed from the floor under the MSF stages and be placed in the intermediate floor between the MSF stages and the VTE effects. This would in effect create a double floor similar to the deaerator structure in the MSF plant, with the ducts formed between the two floors. (See Figure 8.1.3.) Brine leaving an effect would flow through ports in the floor into the longitudinal duct between floors, move longitudinally to the proper stage, and then drop into the transverse duct, which would distribute it across the width of the train. The transverse duct is built against the downstream wall of the stage to which the brine is returning, with the stage wall forming one wall of the duct. In depth, the transverse duct extends from the top of the stage down to near the normal brine level - a depth of more than 6 feet. Orifices are placed at frequent intervals in the bottom of the duct, discharging the returning brine into the MSF brine stream near the downstream end of the stage. Orifices are sized so that the brine level must build up a head of brine in the transverse duct to force the required flow quantities through the openings. This insures a uniform transverse distribution of returning brine, and at the same time retains the feature of self-regulating flow provided by the ORNL bottom - feed brine return design.

Moving the brine return ducts from the floor below the MSF stages to the floor above eliminates the problem of sealing the ducts. Were these ducts to remain below the MSF stages, it would be necessary to provide for vacuum-tight sliding joints between these ducts and the floor slab.

The double-floor construction, in addition to providing ample room for brine return ducts, also provides space for large pressure-equalizing vapor ducts connecting each effect and its corresponding stage. This would reduce the risk of developing a high pressure differential between effects, and might allow the reduction of the safety factor now employed in designing the walls between adjacent effects.

#### Problems Encountered in Design Check of MSF Plant Structure

The MSF evaporator, like the VTE, is basically a very large pressure vessel and, therefore, has design considerations of elevated temperatures and pressures not usually present in concrete structures. In Fluor's opinion, there are unresolved problems with the structure as presented in the ORNL design. These problems are:

- a. Both cold and hot ends are structurally inadequate to resist the horizontal loads derived from differential pressures. Full vacuum conditions were not originally considered by ORNL as a possible design condition.
- b. Water boxes could not tolerate an accidental pressure build-up from pump shut-off head being applied to them.
- c. In the deaerator structure no provision was made for the pressure differential between supply and spray chambers. As a consequence, both ceiling slabs and columns were underdesigned.
- d. The tubular steel framework proposed for the hot end of the plant is not considered suitable for this type of service. Although tubular shapes have less surface area exposed to corrosion than standard rolled shapes, they exhibit, in Fluor's opinion, overriding disadvantages. Their erection costs are much higher than structural members. They are susceptible to corrosion inside the tube an area which is not accessible for inspection.
- e. Differential thermal expansion between the upper structure and the 2-foot-thick floor slab may cause difficulties. The 2-foot-thick MSF floor slab will be heated on only one side. It will expand neither as fast nor as far as the much thinner stage walls, which are heated on both sides. Since these walls are too rigid to deflect, cracks or even failure might occur between duct openings.
- f. The single division wall between trains is impractical. It requires sealed expansion joints at every stage wall and at the roof level. The latter would be likely to shear off from the differential longitudinal expansion of adjacent trains.
- g. There is no positive connection between the 2-foot floor slabs and the 6-inch base slab to transfer horizontal loads to the ground.
- h. There is no provision for drainage of the arched steel roof.
- i. Assuming the ground water table at approximately 4 feet above MLIW, the buoyant forces on the recycle pump pit could float it out of the ground under certain conditions.

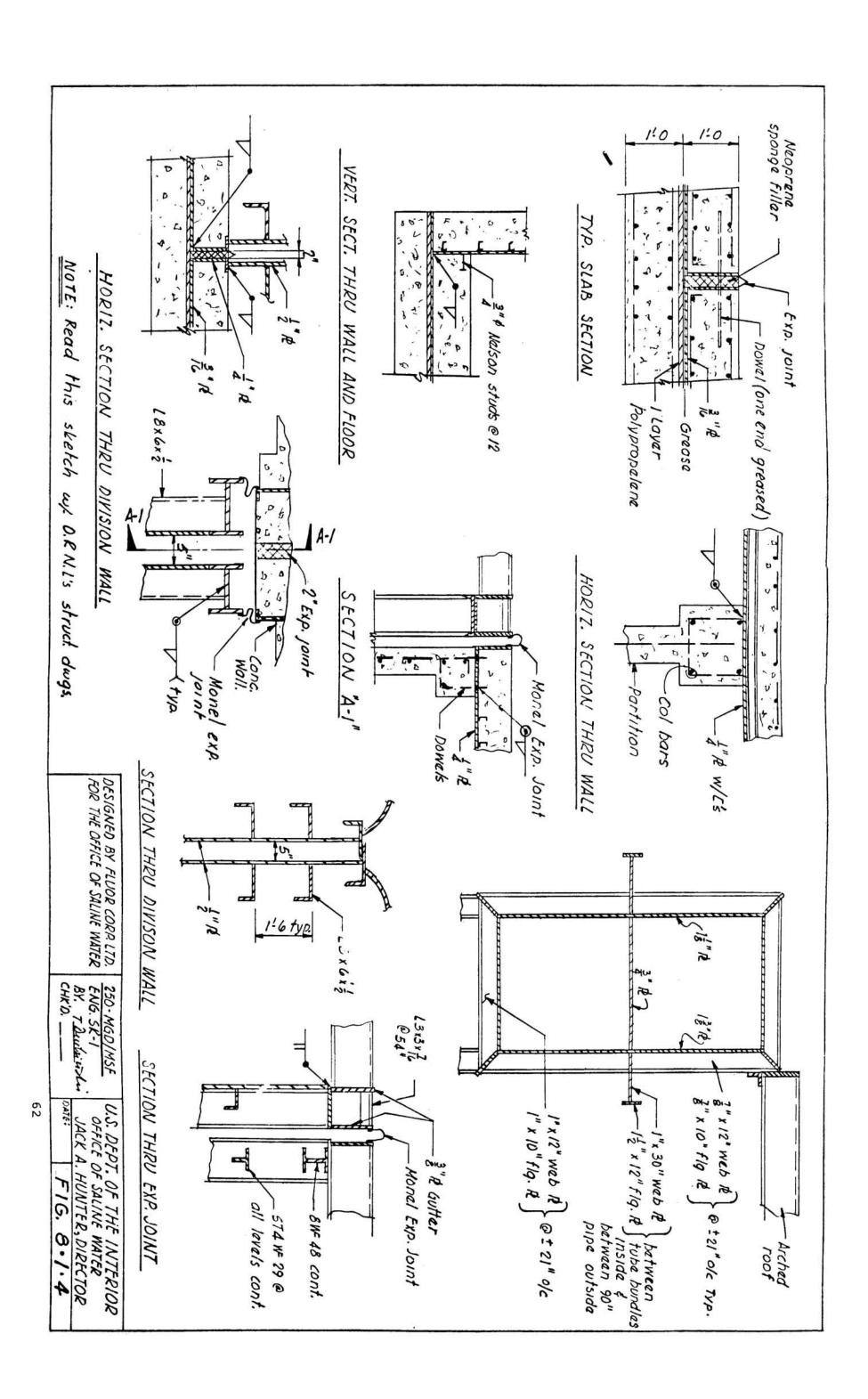
## Changes to the MSF Plant Structure Introduced by Fluor

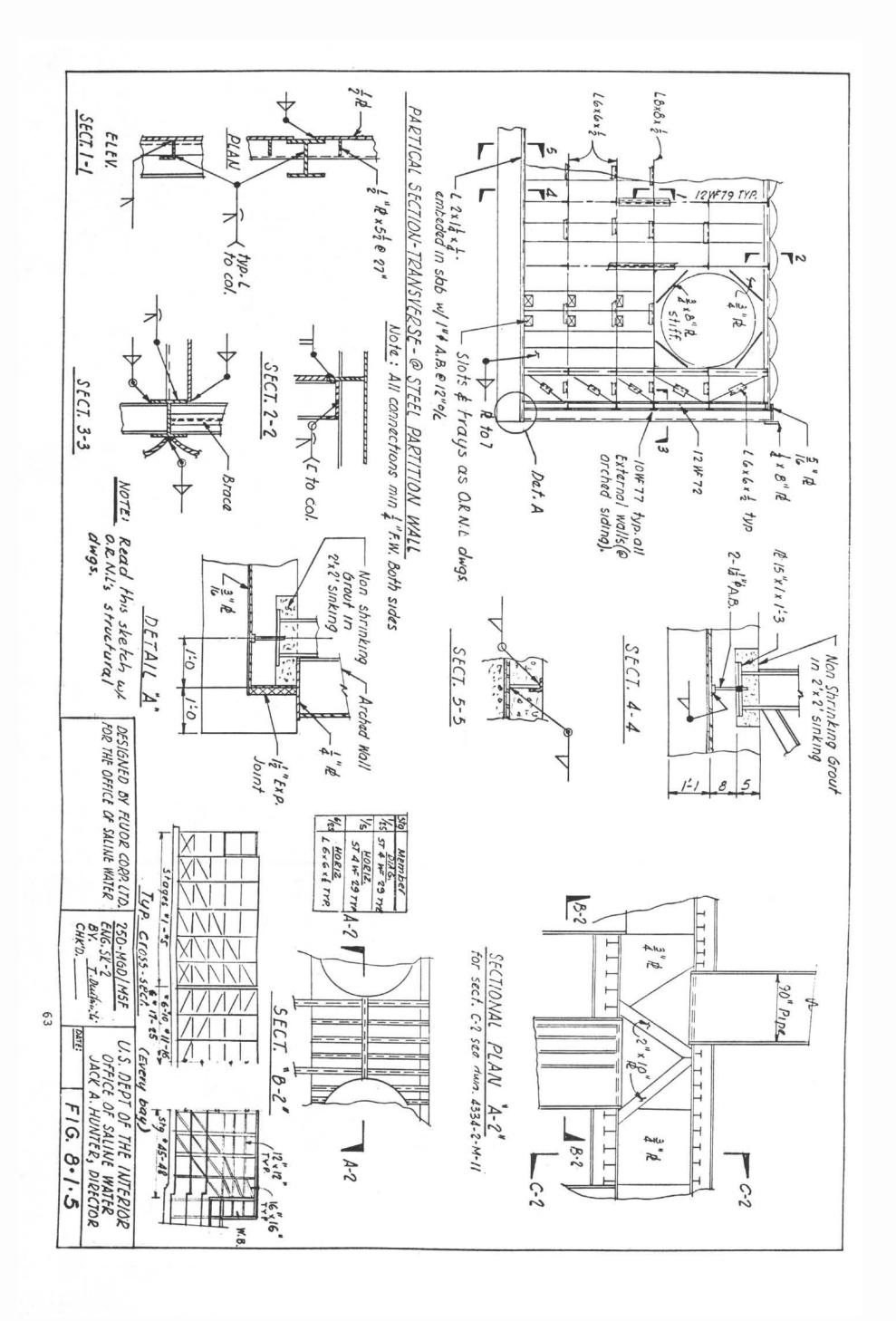
Fluor's cost estimate is based on ORNL's design plus the following changes and additions:

- a. Diagonal struts were added at hot and cold ends to resist horizontal loads.
- b. Water boxes were redesigned. The walls were thickened and horizontal diaphragms added.
- c. The deaerator ceiling and columns were strengthened.
- d. Tubular steel was replaced by standard rolled shapes and the steel structure redesigned.
- e. To reduce the differential thermal expansion between the upper structure and the floor slab, the horizontal joint between slabs was raised to make the thickness of the two slabs approximately equal. This reduces to 13" the thickness of the slab which is cast monolithically with the upper structure and makes its expansion more compatible with that of the stage walls. Also, the 1/4" carbon steel plate originally placed on top of the floor slab was moved to this joint to simplify the construction. Details of the construction and method of maintaining vacuum integrity are shown in Figures 8.1.4 and 8.1.5.
- f. To reduce the number of expansion joints and their inherent weaknesses, a double wall was used between trains.
- g. Shear keys in the shape of crosses were introduced in the floor slab midway between expansion joints in each segment to prevent the upper structure train sliding on the base slab.
- h. Beams, "U"-shaped in cross-section, were welded to the ends of the arched roof at each transverse expansion joint to serve as gutters.
- i. To counteract the buoyancy of the recycle pump pit, the floor slab was extended beyond retaining walls to engage the surrounding soil.

## Recommendations for Design of the MSF Plant Structure

The following changes, while not included in the estimate, could, in Fluor's opinion, improve the structural performance and economy of the MSF plant.





- a. Make columns larger to reduce the required reinforcing steel.
- b. In concrete construction, eliminate roof and stage wall beams and thicken the stage walls where necessary. Also carry the 1/4" roof plate right through the stage walls. This would eliminate many sharp corners, reducing chances of concrete spalling and also reducing the amount of coatings required.
- c. Improve expansion joint sealing method. The 16-gauge monel strip appears inadequate in many cases.
- d. In the horizontal floor slab joint use Teflon pads rather than continuous sheets of polyethylene.

### 8.2 Tube Inlet Nozzles

Most of the operating experience with seawater desalination plants has involved only MSF plants, which, in general, contain no small, critical passageways through which the brine must flow. In contrast, the conceptual design of the VTE plant as proposed by ORNL contains thousands of nozzles, each of which controls the flow of brine into an individual vertical tube.

A limited body of experience with VTE nozzles has been accumulated during operation of the Freeport, Texas pilot plant. Certain of the tube bundles there are fed through nozzles which are similar to those in the ORNL design. Other bundles are fed through slotted weir caps of various designs. Fluor was advised by Freeport personnel that no plugging problems had developed with either configuration. However, experience with the weirs extended over several years, while the nozzles had been in operation only a few weeks. When nozzles are used to distribute brine flow to the vertical tubes, the effect pump completely fills the chamber above the vertical tubes with brine under about 5 psi pressure. A single 1/4-inch tangential orifice in each nozzle meters brine into the tube in a swirling fashion designed to insure complete wetting of the inside tube surface. Part of the brine evaporates inside the tube, and the brine and vapor move cocurrently through the length of the tube. In contrast, when weirs are used to distribute the flow to the vertical tubes, the brine in the brine chamber rises only about an inch above the bottom of the weir slots and the remainder of the brine space is filled with vapor. If a nozzle becomes partially plugged, it will cause scaling in the tube it feeds; if the nozzle becomes completely plugged, or the tube develops excessive scaling, the tube will be effectively removed from service. The possibility of these VTE nozzles becoming plugged with debris from the seawater feed is a matter of some concern to Fluor.

Freeport plant personnel reported to Fluor that the seawater intake there is often loaded with mud and silt, but that sea life is usually minimal. In contrast, Southern California coastal waters contain heavy concentrations of sea plants and animals, especially at certain seasons of the year -- much more than was observed in the intake channels and on the traveling screens of Freeport or other plants. One of the sea plants frequently observed on the screens at the San Diego Test Center resembles heavy black horsehairs. Some of these would doubtless pass through or around the intake screens and might also get through the deaerator. If this occurred, they would in time be expected to partially or completely plug the 1/4-inch orifices in the VTE nozzles.

Despite the apparent lack of problems encountered at Freeport, Fluor believes that the VTE tube inlet devices could become a major source of trouble for such a plant located in Southern California. This applies particularly to nozzles, but to a lesser extent to weirs as well. Fluor believes that for a Southern California location it will be necessary to install a finer mesh intake screen than is customarily used on MSF plants.

### 8.2 Tube Inlet Nozzles (Cont'd.)

A mesh opening of 1/8" is recommended, in contrast to the 3/8" to 1/2" mesh commonly used in MSF plants. To prevent excessive pressure drop through a finer mesh screen, it will have to be rotated and backwashed more frequently than is usually practiced.

Certain sea life--notably mussels and barnacles-- pass through a free-swimming larval stage before they affix themselves to a suitable solid surface. In this larval stage they can readily move through even fine screens, attach themselves to walls on conduits, and continue their growth within the inlet portions of the plant. When they are later dislodged, their shells can provide debris large enough to cause serious fouling of brine flow channels. It is possible that adequate control of these animals may require a more vigorous chlorination schedule in the VTE plant than for the MSF plant. Adequate control would require that they be killed before they can grow to harmful size. It is expected that more definite information will be available before a large plant is built. Operation of the proposed VTEX plant at the San Diego Test Center should reveal the extent of this potential problem.

### 8.3 Deaerators

Direct comparisons between the VTE and MSF deaerators could not be made because the vapor source and operating temperatures are not the same; however, a comparison of deaeration area allotted per gpm treated, showed them to be substantially the same.

In the VTE plant, deaerator nozzles are made of Saran plastic, and are connected to pipe headers mounted at the top of the deaerating space (see Drawing V-7). Each nozzle is designed for a flow of 500 gpm, and there are a total of 580 nozzles in the plant - 145 per train. There are 45 nozzles (per train) in the hot section of the deaerator, and 100 nozzles in the cold section. A portion of the plant heat reject water at 104 F is acidified and flashed in the hot section, cooling itself to 77 F and deaerating itself in the process. Steam from the hot side of the deaerator is guided to the cold side, where it flows countercurrently to the acidified 65 F seawater spray, stripping the dissolved gases from it and warming it to 77 F. Two packed beds in series assist in providing adequate surface area for efficient deaeration. Carbon dioxide, formed by the action of acid on the seawater feed, is also removed.

The VTE plant deaerator is an untried design requiring further testing. It needs no external steam for stripping, and rejects a minimum of steam to the external barometric condensers, but will require additional development work to prove its performance.

The MSF plant design also provides for Saran deaerator nozzles, but the method of installation is different from that of the VTE plant. Deaerator feed water is pumped to reinforced concrete water boxes in each deaerator section, and the full-cone spray nozzles are mounted directly beneath the water boxes, eliminating internal piping within the deaerator. Each nozzle is rated at 1000 gpm at a differential pressure of 5 psi, and each train utilizes 172 nozzles, for a total of 688 nozzles in the entire plant.

The deaerator employs two-stage spray-nozzle type deaeration, with the water from the first stage collected and pumped to a second set of nozzles. The first stage, in turn, is divided into two sections. The first section, containing 14 nozzles in each train, introduces cold, acidified seawater at 65 F. The second section, containing 72 nozzles, feeds acidified seawater which has been warmed to 92 F in the heat reject section of the evaporator. These two streams are combined and sprayed into the second stage of the deaerator through 86 nozzles per train. Stripping steam from the final heat reject stage of the evaporator is introduced to the deaerator to assist in the deaeration and decarbonation of the feed stream.

### 8.3 Deaerators (Cont'd.)

The only significant and controllable difference between the dearrators of the two plants is in the internals of the dearrator itself. The VTE plant uses packing while the MSF plant relies on spray nozzles only. Since the VTE represents the more advanced technology and is the more conservative design, it was necessary to allow for the cost of packing in the MSF plant in order to make the costs truly comparable.

### 8.0 - 250 MGD CONCEPTUAL DESIGN - VTE AND MSF PLANTS

### 8.4 Pumps and Drivers

In the current study, Fluor's principal objective with respect to pumps and drivers was to be sure that pumps were provided for each required service, that they were of suitable type and size, and that they were chosen on a comparable basis for the two plants. Then prospective vendors were contacted for pricing information.

The required capacity for each pump was determined from the process flow diagram or ORNL computer printout for each stream. To arrive at the pump design flow, this quantity was then increased by an arbitrary 2.5% for all except the VTE effect pumps. This increase was to compensate for decreasing pump performance from impeller and casing wear during the period between pump overhauls. Pump heads used were those specified on the computer printout, without any contingency added. Pump driver horsepower requirements were calculated from flow and head requirements, using typical pump efficiencies derived from Bureau of Reclamation studies. This figure for required pump horsepower then became the basis for driver selection, and the next larger standard electric motor size was specified.

It was noted that, for the VTE reference design, in each instance the efficiencies calculated from the computer printouts were higher than those found using the Bureau of Reclamation procedure. In contrast, the efficiencies implicit in the MSF reference design, as calculated from the computer printouts, were equal to or lower than those based on Bureau of Reclamation procedure. This would tend to make the pump power consumption somewhat low for the VTE plant and slightly high for the MSF.

Although the VTE pump efficiencies were decreased, and capacities increased, from those used in the reference designs, the latest ORNL computer printout on the VTE plant showed substantially lower head requirements for the booster and the make-up pumps. This resulted in a net effect of a very slight reduction in power requirements for the VTE plant but a substantial reduction for the MSF plant.

Vertical pumps were selected for all of the major services in both plants. Many pumps were necessarily located in pits because of NPSH requirements - a configuration well-suited for vertical pumps. The pit can be smaller than that required for a horizontal pump, and the motor can be located at ground level or in the pit as desired. Vertical pumps are also adaptible to either wet- or dry-pit construction, while horizontal pumps are not. Engineers and operators differ as to their relative merit. Both types of pump are in service at St. Thomas, and the vertical pumps are preferred. The process engineer at St. Croix also preferred vertical pumps, citing savings in foundation costs and reduced downtime and maintenance costs when compared to horizontal pumps.

<sup>1</sup> Research on Pumping Unit Studies Final Evaluation Report, U.S. Bureau of Reclamation Research and Development Progress Report No. 205, September, 1966

### 8.4 Pumps and Drivers (Cont'd.)

Dry pit construction was chosen for most services for both plants, with the wet-pit VTE effect pumps being the major exception.

Installed spares were provided for the small pumps used to add chemicals to the make-up streams. The chemicals are necessary for plant operation, and the cost of spares for small pumps is small. For the larger pumps, however, the cost of spare pumps would be substantial, and would be more difficult to justify. For this study, therefore, it was decided to provide no spares for any of the large pumps, with the exception of the VTE effect pumps. These pumps must be considered in a different category from other pumps in the two plants for the following reasons:

- a. Aside from the effect pumps, the complement of pumps in the VTE and MSF plants is quite similar 20 major pumps in each plant. In addition to these, however, the VTE plant has 60 effect pumps. Because of these extra pumps, the VTE plant is expected to require more maintenance in this area than the MSF plant.
- b. Many of the effect pumps are identical, so that one spare pump and motor could serve for a number of pumps.
- c. The effect pumps are moderately sized (350 to 500 hp) and priced, so that the cost of warehouse spares would not be excessive.

Considering the above factors, it was decided that an equitable solution would be to charge the VTE plant with warehouse spares in order to equalize the reliability of two plants. Accordingly, one warehouse spare pump and motor was purchased for the 4 Effect I pumps, one for the 4 Effect II pumps, two for the 32 Effect III through X pumps, and one for the 20 Effect XI through XV pumps.

A complete tabulation of pumps and drivers is included in the equipment list for each plant - section 9.1 for the VTE plant and section 10.1 for the MSF. These pump tables also include the major performance specifications of pumps and motors, type of pump, and materials of construction. Additional comment on certain of these pumps is included below.

The design flow rate of the VTE booster pumps was increased to correspond to the process flow sheet.

The seawater intake pump in the MSF plant did not have sufficient head to feed the barometric condensers. Four (one for each train) booster pumps were added to increase the head to the condensers.

No provision had been made by ORNL in the VTE plant for transferring barometric condenser effluent from the seal pit to the reject header. Two small pumps were added for this purpose.

### 8.4 Pumps and Drivers (Cont'd.)

Although the ground rules in the reference design for the VTE plant specified using synchronous motors for motors larger than 450 hp, Fluor has specified induction motors for the 500-hp Effect I pumps. This was done in the interest of uniformity and simplicity, since all other effect pump motors are induction type.

The geometry of the suction conduit leading to the MSF brine recycle pumps was not adequately designed, in Fluor's opinion. This was modified to provide a more smoothly contoured low-velocity entrance region, with provision to minimize vortexing and vapor entrainment. This is discussed in more detail in Section 3.1.

### 8.0 - 250 MGD CONCEPTUAL DESIGN - VTE AND MSF PLANTS

### 8.5 Chemical Systems

### General

Continued smooth and proper operation of both VTE and MSF plants is dependent upon the addition of small amounts of chemicals at the proper times and places, and in suitable amounts. The purposes of these additions include inhibiting marine growth, preventing fouling of heat transfer surfaces, preventing corrosion, and eleminating brine foaming. The several chemical systems are described separately in the following paragraphs, and are generally similar for both plants. Details of the chemical addition systems are shown in the P&I Flow Diagrams - Drawing No. V-4 for the VTE plant and No. M-4 for the MSF.

### Chlorination

Raw seawater abounds with plants and animals, both living and dead. If no preventive measures were taken, the intake structures of desalination plants would soon become heavily populated with mussels, barnacles, and other forms of sea life. This would eventially interfere with the normal flow of water. Even more important, the shells and debris from animals growing within the plant would foul the plant internals, such as exchanger tubes, nozzles, and deaerators.

Chlorination of the seawater feed is the best method of control in most cases. The quantity of chlorine needed varies with locale and with ocean temperature, but 5 to 8 parts per million is the usual requirement. The free chlorine oxidizes organic matter in the feed water, and is consumed in the process. Chlorine requirements are usually gauged by measuring the amount of residual chlorine still remaining after the immediate demands of the feed water have been met. Sufficient chlorine is added to leave a residual of, say, 0.5 to 0.75 ppm. This is enough to kill most marine life, or at least inhibit its growth. The chlorine is injected at the intake structure, at the extreme outer end of the submarine intake conduit. Adding the chlorine at this point controls marine growth throughout the entire intake system.

Chlorine is shipped to the site as a liquified gas in 55-ton tank cars. The liquid chlorine is transferred from the tank car by its own vapor pressure, and is then vaporized in steam heat exchangers. The vapor is given 15-20 F superheat to prevent possible condensation in the line. Water jet exhausters mix the vapor with a side stream of seawater from the main intake pump, diluting it with about 500 parts of water. A centrifugal pump then forces this heavily chlorinated stream to the submerged intake structure where it blends with the main intake stream. An installed spare pump is provided to insure against pump breakdown. The chlorination system is capable of continuously adding 20 ppm chlorine to the feed stream if required. It is anticipated that about one-third of this amount will be sufficient, and that intermittent rather than continuous chlorination may accomplish the purpose. Some operators, however, report that mussels can tolerate intermittent chlorination by simply closing their shells and waiting until the chlorine is gone. If this is true, continuous chlorination will probably be required, and money has been provided for operation on this basis.

### 8.5 Chemical System (Cont'd.)

### Acid Addition

Untreated seawater will begin to precipitate alkaline scale on heat transfer surfaces at temperatures well below 200 F. With proper acid treatment, evaporation temperatures can be raised to the 250-260 F. range without encountering scale deposits. Sulfuric acid, least expensive of the mineral acids, is commonly added to the make-up water of desalination plants in an amount sufficient to prevent deposition of calcium carbonate and magnesium hydroxide. The exact amount required is determined most accurately by pH measurement of the treated stream, but is usually about 120 ppm for seawater. This will require about 225 tons/day of acid for the VTE plant, and about 275 tons/day for the MSF. Sulfuric acid at 93% concentration will be delivered to the site by rail shipment and stored in a 500,000-gal tank - sufficient for 10-14 days normal plant operation.

The ORNL designs proposed the erection of an on-site captive acid plant, and showed a substantial cost savings over the purchase of acid from existing suppliers. It is not likely that permission could be secured from air pollution control authorities to erect a new acid plant in smog-conscious Southern California. Since the purpose of the current study is to provide an equitable comparison between VTE and MSF plants, it was agreed that Fluor would not include an acid plant in the plans, but would provide for purchase of acid from existing manufacturers.

Acid is supplied to the make-up water of each train by a seal-less (canned) centrifugal pump, whose pumping rate is automatically proportioned to the flow rates of make-up water. The proportioning ratio is manually adjusted as necessary to maintain the proper pH in the make-up stream. The acid is injected into the central flow of the conduit through six circumferentially-spaced Hastelloy B nozzles. Since the continuous addition of acid is essential for plant operation, an installed spare pump is provided for each train, with alarms to warn the operator if the feed pH drifts outside its prescribed limits.

### Caustic Addition

Provision is made for the addition of caustic if it is required to prevent corrosion of the brine heater tubes. When sulfuric acid is added to the make-up water, it normally drives the pH down to the range of 4.5 to 5.5. The carbonate ions are broken down, forming carbon dioxide, which is dissolved in the water. In the deaerator most of the carbon dioxide is removed, along with other dissolved gases. The elimination of the carbon dioxide usually raises the pH of the water enough so that it is no longer corrosive to the heat exchanger metals. However, if the pH of the make-up water is still too low, sodium hydroxide will be added as required to adjust the pH to approximately 7.0 to 7.5.

### 8.5 <u>Chemical System</u> (Cont'd.)

Sodium hydroxide is delivered in carload lots as a 50% solution, and stored in a 50,000-gal supply tank. It is added to the make-up water at a point downstream of the deaerator, using a single canned centrifugal pump for the entire plant. Feed rate for each train is adjusted automatically in proportion to recycle flow rate, with reset from a pH sensing probe in the recycle line. An installed spare pump is provided to insure against breakdowns.

### Anti-foam Addition

Although foaming is not always a problem in desalination plants, it has occurred at Point Loma when an anti-foaming agent is not used. Provision has been made in this study for the addition of 0.5 ppm of Hagen C-l anti-foaming agent to the make-up water of each plant. The concentrated chemical, delivered in drums, is diluted to a 10% solution for ease in metering the flow to the plant make-up stream. Addition rate of the 10% solution is about 2 gpm for the entire plant. The 10,000-gal storage tank is provided with a mixer to facilitate blending the stock solution. A single centrifugal pump (with installed spare) provides anti-foam solution for the entire plant. The feed rate to each train is proportioned from the flow rate in the make-up stream.

### Sodium Sulfite and Ferrous Sulfate Addition

Provision has been made for the addition of small amounts of two other chemicals to the system. The deaerator will remove the major portion of the dissolved oxygen in the make-up water, down to perhaps 50 ppb. To scavenge the remaining oxygen, and to react with any oxygen which leaks through the evaporator walls, the addition of 1 or 2 ppm sodium sulfite to the feed is suggested. The injection of 5 ppm ferrous sulfate on an intermittent basis is also suggested to provide a corrosion inhibiting film and reduce the incidence of tubeside corrosion. Each of these chemicals is stored as a solution of 10 or 15% concentration and is added to the make-up feed with a small centrifugal pump. Addition rates to each train are proportioned from the flow rates in the respective make-up streams.

### 8.0 - 250 MGD CONCEPTUAL DESIGN - VTE AND MSF PLANTS

### 8.6 Instrumentation

### General

The instrumentation proposed for both the VTE and MSF plants is essentially in accordance with that proposed in the ORNL conceptual designs, except as noted in the following comments and discussion. For pricing purposes, instrumentation design and specifications are in accordance with Fluor standards.

A comparison of the instrument application diagrams of the VTE and MSF plants indicated a difference in concept in the instrumentation recommended. Fluor believes the MSF plant should be instrumented in as detailed a manner as the VTE plant. Instrumentation such as pressure, level, and temperature indication and recording, which Fluor believes necessary for efficient operation, have been added to the MSF plant. Alarms have been added where deemed necessary for warning of approaching abnormal operating conditions. In general, the instrumentation has been upgraded to the same concept of design as the VTE plant. A brief description of instrument systems follows:

### Temperature Instruments

Conventional analog temperature recorders and controllers were applied as required for general plant operation.

Fluor would propose the use of standard electronic indicators, with selector switches, such that the operator could look at any one trouble point, or scan all points.

### Thermocouples

Thermocouples used in the flash stages are a design using as a thermowell 316 S.S. tubing with closed end. These will be inserted through the concrete top of the flash stages, internally supported for mechanical strength, and sealed to prevent air leaking into the stages. Elements are single-sheath type T (copper-constantan) thermocouples.

### 8.6 Instrumentation (Cont'd.)

EMF-current converters are used to convert millivolt thermocouple signals to current signals for recorders and controllers.

### Pressure Transmitters

Although ORNL specified 316 S.S. trim for the pressure transmitters, Fluor believes that monel would be more resistant to seawater exposure in areas not subject to contamination by hydrogen sulfide. A Southern California location meets these requirements; accordingly, Fluor has specified monel bodies and diaphragms for the pressure transmitters.

### Differential Pressure and Flow Transmitters

Standard force-balance type D/P cells for all except chlorine service are specified with monel bodies and diaphragms, for reasons explained in the previous paragraph. Cells handling dry chlorine gas have carbon steel trim.

### Flow Tubes

Insert type plastic "low head loss" flow tubes have been selected for flow measurement in pipe sizes up to 60". For pipe sizes over 60" tubes of plastic-coated carbon steel have been specified. These devices present a minimum of pressure drop to the system and with factory calibration the element can be guaranteed to an accuracy of 1/4%.

### Liquid Levels

Liquid level probes in the brine stream of each stage will transmit a signal to the control room, using capacitance type probes for level measurement. Indication only will be provided in the control room. For the upper brine trays in the MSF plant, the arrangement of the level-sensing probes is based on the presence of cross troughs to interconnect brine trays which lie on the same elevation. These cross troughs, located at about every eighth stage, will maintain brine at approximately the same level in all trays, so that one measurement in each stage will suffice. Local indication of brine level is by means of a gauge glass in each stage.

### Control Valves

Hydraulically operated butterfly valves are specified in the ORNL conceptual designs; however, Fluor believes that pneumatically-operated valves with positioners are satisfactory and will provide a substantial cost reduction. The valves are, therefore, designed to operate on 100-125 psi instrument air, with a standard 3-15 psi pneumatic positioner controling a pneumatic cyclinder. This arrangement will provide the valve torque, speed, and stability necessary for good control. Standard-shaped butterfly discs are specified here, but for a final plant design, airfoil- or fishtail- shaped discs should be investigated. These show a 30% reduction in operating torque requirements in sizes up to 36-inch.

### 8.6 Instrumentation (Cont'd.)

Manufacturers are confident that this would also be true for large valves, but have no test information at this time. These streamlined discs would be advantageous for normal throttling service, and could represent a substantial saving on capital cost.

Butterfly type control valves with Ni-resist discs and bodies, 316 S.S. shafts, and Nordel tight-shutoff seats are specified. Control valves in the chemical additive systems are specified with monel bodies and trim, and no positioners.

### Relief Valves

Fluor assumes that adequate pressure relief is furnished in the steam supply system and has not included this item in the water plant. Brine piping is designed for full pump shutoff head, eliminating the need for relief valves.

### Switches, Alarms, and Solenoid Valves

Dual function milliamp relays with adjustable contacts are specified for alarm and shutdown. Switches and/or relays are mounted behind the control panel.

Annunciator alarm panels are solid state with backlighted nameplates and with Test and Acknowledge buttons. Switch contacts open to sound alarm, Annunciator sequence: alarm or abnormal condition - light and horn sounding; after acknowledgment, light steady and horn silent; and upon return to normal, light out and horn silent. Each panel is specified with 20% spare annunciator points.

### Control Panel and Analyzers

The control panel is a console type using high density instruments, with three rows of miniature instruments in the vertical face and two rows on the inclined face. Shut-down, touch temperature panel, alarm test buttons, and hand switches are mounted on the board.

A console desk is provided for the free-standing logging typewriter. The desk also contains a touch temperature panel, manual shutdown switches, and hand switches. These switches are for ease of operation, and are connected in parallel with those on the panel.

Analyzers are furnished as a package unit for field installation and indication. They transmit a compatible signal to a board recorder.

### Panel Instruments

Recorders and recorder-controllers are miniature electronic type suitable for high density mounting. Adjustments and servicing are performed from the front of the panel. Indicators are vertical scale milliameter type for high density mounting.

### 8.6 <u>Instrumentation</u> (Cont'd.)

### Miscellaneous

Electro-pneumatic transducers receive an electronic signal from the board controller and convert it to a 3-15 psi pneumatic signal to the control valve. They are suitable for field mounting on a floor stand.

The digital data system logs on a time sequence or on demand. A free-standing typewriter is provided for recording data. The data system is capable of logging 200 thermocouple temperature points and 150 pressure points. Logging is accomplished without upsetting any of the control systems.

### 8.0 - 250 MGD CONCEPTUAL DESIGN - VTE AND MSF PLANTS

### 8.7 Electrical

Primary power is supplied to the evaporator plant from the auxiliary bus in the main power plant switchyard. A radial feed system is used, with a single feeder supplying one train as shown on the Electrical One-Line Diagrams, Drawings V-10 and M-9 for the VTE and MSF plants, respectively.

In the reference designs, motors smaller than 500 hp in the VTE plant, and smaller than 600 hp in the MSF plant, were supplied from 480 volt lines. Aside from fractional horsepower motors, the MSF plant has relatively few motors in this size category - 16 motors with 3500 total connected horsepower. In contrast, the VTE plant has 76 such motors with 26,000 total connected horsepower. There would be a significant economic penalty incurred in the VTE plant in providing a 3 phase, 480 volt distribution system of this magnitude. Accordingly, Fluor has changed the secondary distribution system to 2400 volts in both plants. This reduces the electrical capital cost on the VTE plant by approximately \$750,000, and results in a small saving on the MSF plant.

The four 17,000 hp motors on the brine recycle pumps in the MSF plant are provided with 13,800 volt 3 phase power from individual captive transformers. All other motors in both plants, 100 hp or larger, are supplied with 2400 volt 3 phase power. Integral horsepower motors smaller than 100 hp are provided with 480 volt 3 phase power, and 208/120 volt single phase power is available for fractional horsepower motors, control, and lighting.

Motors 600 hp and larger are synchronous type, unity power factor, with brushless exciters. Motors smaller than 600 hp are induction type.

Motors 600 hp and larger have weatherproof enclosure and are water-to-air cooled. Smaller motors are epoxy encapsulated weatherproof or totally enclosed fan cooled.

Transformers and substations are outdoor type with protection suitable for seacoast atmosphere.

Underground feeders are provided. Although the initial cost of overhead distribution is less than that for underground, the underground is relatively unaffected by weather, provides greater reliability, and is less susceptible to atmospheric contamination. Underground ducts also facilitate evaporator maintenance, since there are no overhead lines on or around the evaporator plant.

Duct banks of polyvinyl chloride conduits embedded in concrete are provided for the underground cables. Polyethylene insulation is used for 2400 and higher voltage, and THW insulation is used for lower voltages.

### 8.7 Electrical (Cont'd.)

All motors for the evaporator plant are remotely controlled from the control room. Underground ducts carry the control, communication, and instrument cables to the control room. Adequate alarms, metering, and indication are provided in the control room for remote operation.

Mercury vapor lighting fixtures mounted on steel poles provide lighting for streets and outdoor operating areas. A manual fire alarm system is integrated into the total plant fire alarm. The telephone system is of a commercial type.

A copper ground mat is provided which will be tied into the switchyard and power plant grid. All exposed metal will be connected to this ground system.

The total estimated load for the VTE plant is 36 mw. The estimated load for the pump motors of each train is 8.75 mw. The chlorination system and other auxiliary loads are estimated at 1 mw for the total plant.

The total estimated load for the MSF plant is 61 mw. The estimated load for the pump motors of each train is 15 mw. The chlorination system and other auxiliary loads are estimated at 1 mw for the total plant.

### 8.0 - 250 MGD CONCEPTUAL DESIGN - VTE AND MSF PLANTS

### 8.8 Material Selection

The materials of construction and protective coatings included for the major portions of both the VTE and MSF evaporator buildings are shown in Table 8.8. The selection of materials is in some instances different in one plant than in the other. This is in some cases due to the requirements or limitations of the plants. For example, more care is taken in the VTE plant than in the MSF to exclude corrosion products from the brine stream because of the danger of plugging the small holes in the distribution nozzles at the top of each tube in the vertical tube effects. In other cases the differences in constructural materials chosen reflects a difference in the philosophy of the designers at the time the plants were initially designed. As an example, when the MSF plant concept was developed, it was considered best to limit the use of concrete to temperatures below 200 F. However, when the VTE plant was being designed, concrete was used up to 240 F, and, with a polypropylene or steel liner, up to 260 F. This represents a "change in the rules," and renders the plant comparison more difficult. In the current study, the plants were placed on an equal basis where this was possible. If this represented a major design change which was beyond the scope of this study, no changes were made, but the different assumptions were noted and the probable effects discussed.

The first four entries in Table 8.8 represent the ground rules for the specific materials choices noted in the later items. The philosophy they represent may be summarized as follows:

- a. Condensing vapor and distilled water will attack concrete; therefore, the concrete must be protected in vapor areas and product trays.
- b. Carbon steel will be corroded at a slow rate by vapor or brine. Therefore, in the MSF plant allow 1/8" corrosion allowance. In the VTE plant, allow 1/8" corrosion allowance, and in addition cover with polypropylene sheet to prevent corrosion products from getting into the brine and plugging the nozzles on the vertical tubes.
- c. Concrete is limited as to the temperatures it can stand while in contact with brine, and must be protected if these temperatures are exceeded. In the MSF plant, concrete construction is limited to temperatures below 200 F; in the VTE plant, concrete is used for brine up to 260 F, but the surface is protected with polypropylene or steel lining above 240 F.
- d. The exterior boundaries of the evaporator buildings (walls, floor, roof) must be steel-lined to provide a continuous barrier against the in-leakage of air.

8.8 Material Selection (Cont'd.)

TABLE 8.8

# STRUCTURAL MATERIALS AND PROTECTIVE COVERINGS EMPLOYED IN THE VIE AND MSF DESALINATION PLANTS

ve Coverings	MSF Plant	Concrete is not used above 200 F; unlined below 200 F	PP covered	Not protected by any lining or coating, but 1/8" corrosion allowance (CA) is provided.	Must contain steel membrane to prevent air inleakage (i.e., they are either built of steel, or are steel-lined)	Includes 1/8" CA; unlined	Steel lining includes 1/8" CA	Includes 1/8" CA
Corrosion Allowance and Protective Coverings on Structural Elements	VTE Plant	Covered with 1/8" polypropylene (PP) sheet above 240 F; unlined below 240 F	PP covered	PP covered (to prevent corrosion products getting into brine or condensate)	Must contain steel membrane to prevent air inleakage (i.e., they are either built of steel, or are steel-lined)	Included 1/8" CA and PP cover	Not applicable. (All roof on VTE plant is steel.)	Includes $1/8$ " CA and PP cover
Basic Structural Material		Concrete exposed to liquid brine only	Concrete exposed to vapor, or to liquid condensate	Carbon steel on walls on roof, exposed to vapor or brine	Exterior walls, roof, and floor of evaporator buildings	Steel	Concrete, with $1/\mu$ " steel lining	Concrete, with $1/4$ " steel lining
Structure		General	General	General	General	Roof	Roof	Outside walls

CA = Corrosion Allowance PP = Polypropylene

### 8.8 Material Selection (Cont'd.)

TABLE 8.8 (Cont'd.)

### STRUCTURAL MATERIALS AND PROTECTIVE COVERINGS

Structure	Basic Structural <u>Material</u>	Corrosion Allowance and Protective Coverings on Structural Elements	ive Coverings nts
		VTE Plant	MSF Plant
MSF ceiling (VTE Plant)	Concrete	PP covered	Not applicable
Partitions between stages or effects (in vapor space)	Concrete	PP covered	PP covered
MSF floor	Concrete, with 1/4" steel lining	PP covered (See Note 1)	Epoxy-phenolic coating
Beams, columns, and load-bearing walls (in vapor space)	Concrete	PP covered	PP covered
Brine trays (where exposed to vapor)	Concrete	PP covered	PP covered
Brine trays (where exposed only to brine)	Concrete	Unlined	Unlined
Brine trays	Steel	Not applicable. (No steel brine trays in VTE.)	Epoxy-phenolic coated all over (See Note 2)
Product trays	Concrete	PP covered	PP covered
Product tra <b>y</b> s	Steel	No steel product trays	Epoxy-phenolic coated all over (See Note 2)

Note 1 - Fluor's estimate is for epoxy-phenolic coating Note 2 - Fluor's estimate includes only a 1/8" corrosion allowance

8.8 Material Selection (Cont'd.)

TABLE 8.8 (Cont'd.)

### STRUCTURAL MATERIALS AND PROTECTIVE COVERINGS

otective Coverings Elements	MSF Plant	Not applicable	Not applicable	Not applicable	Not applicable	Includes $1/8$ " CA on steel; no other covering	Not applicable. MSF deaerator roof is concrete	Includes $1/8$ " CA on steel; $1/2$ " Lumnite cement coated	Includes $1/8$ " CA on steel; $1/2$ " Lummite cement coated	1/2" Lumnite cement coating	Coated with 1/2" Lummite cement	Lined with 1/2" Lummite cement, outside uncoated
Corrosion Allowance and Protective on Structural Elements	VTE Plant	Includes 1/8" CA on steel; coated with 1/2" Lummite cement	PP covered	PP covered	PP covered	Includes 1/8" CA on steel; PP covered	Includes $1/8$ " CA on steel; $1/2$ " Lummite cement coated	Not applicable. VTE de- aerator roof is steel.	Includes $1/8$ " CA on steel; $1/2$ " Lummite cement coated	1/2" Lummite cement coating	Coated with 1/2" Lumnite cement	Lined with 1/2" Lummite cement, outside coated with epoxy-phenolic
Basic Structural Material		Concrete, with $1/4$ " steel lining	Concrete	Concrete	Cast-in-place concrete	Concrete with $1/4$ " steel lining	Steel	Concrete, with $1/4$ " steel lining	Concrete with $1/4$ " steel lining	Concrete	Steel and concrete	Steel
Structure		Wall, roof, and floor of VTE Effect I	Floor of VTE Effects II and III	Floor of VTE Effects IV thru XV	Brine ducts and vapor ducts	Wet pits for pumps	Deaerator roof	Deaerator roof	Deaerator walls, external	Deaerator walls, internal, and floor (this is an internal floor)	Deaerator internals	Deaerator piping

### 8.8 Material Selection (Cont'd.)

Interior boundaries between evaporator areas with more than 2 or 3 psi differential pressure between them must be lined on the high-pressure side if lining is to be used on the low-pressure side. For example, the ceilings in the MSF stages are polypropylene lined to protect the concrete in the vapor phase. The stage ceiling also serves as the floor for the effect directly above, which operates at a higher pressure. To prevent water from seeping through the concrete and separating the polypropylene liner from the stage ceiling, a polypropylene liner is also required on the effect floor.

Polypropylene sheet has been specified as a protective covering for concrete and steel in many portions of both VTE and MSF plants. It is reported to have withstood extended life tests in an unloaded condition while submerged in 265 F brine. Since gradual creep under continued stress is a common mode of failure for thermoplastics, further tests under loaded conditions will be necessary before extensive use of polypropylene in this environment. For the purposes of this study, sufficient money has been provided to install polypropylene sheet in the plant areas indicated. It is assumed that by the time such a plant is built, continuing tests will have verified the suitability of polypropylene or some other more appropriate material of comparable cost. The reference design specified the use of all-polypropylene stage walls for Stages 34 through 50 in the VTE plant. Because of the creep characteristics of polypropylene at elevated temperatures, there is serious doubt as to its suitability as structural members. Fluor has therefore substituted concrete for the polypropylene walls, and has allowed for this in the cost estimate. Sheet 1 of 8

### EQUIPMENT LIST 9.1

### FOR

### 250 MGD VERTICAL TUBE EVAPORATOR (VTE) PLANT

rotal No. Required In Plant

4

### Evaporators and Exchangers

Evaporator Vessels (buildings). (Each building contains one train)

Concrete with steel liner and steel roof sections.

and cold ends, respectively; 33' to 374' long; 64' and 118' wide at Trapezoidal shape

hot

Each evaporator vessel contains several integral parts, separately described below. Quantities shown are for a complete 250 MGD plant, unless otherwise indicated.

35' high

60 total (15 each train) 6,113,110 ft.<sup>2</sup> total Vertical\_Tube Evaporator Effects . п

Vertical tube heat transfer area Number of vertical tubes

Tube description

611,311 total 3" o.d. double-fluted, 0.049" developed wall thickness, 10' long, CDA Alloy 194 (98.68% Cu, 2.3% Fe, 0.02% Phosphorous)

200 total (50 each train)

Flash Evaporator stages ٠ م

Feedheater (stages 1 - 50) Final brine heater stage Horizontal tube data:

Number of tube bundles Number of tubes

Tube material Tube surface Tube size

3,996,937 ft.<sup>2</sup> total area 236,260 ft.<sup>2</sup> total area 8 total (2 each train) 58,017 total (7252 each bundle) 3/4" o.d. x 0.035" wall x 371.6' long

Smooth (no extended surfaces)

90-10 Cu-Ni alloy

### Sheet 2 of 8

## 9.1 EQUIPMENT LIST FOR 250 MGD VTE PLANT (Contd.)

Total No. Required In Plant

Evaporators and Exchangers (Cont'd.)

### Heat Rejection Tubes ့်

Number of tubes Tube material Tube surface Tube size Tube area

643,081 ft.<sup>2</sup> total (160,770 ft.<sup>2</sup> per train) 27,716 total (6929 per train) 1" o.d. x 0.049" wall x 88.6' long Smooth

90-10 Cu-Ni

with counter-current stripping and double-packed bed. No external steam used

580 total (145 per train) 500 gpm each Saran

Two-section flash co-current deaerators

### Deaerators ф

Description

Number of spray nozzles Spray nozzle material Spray nozzle size

Vapor Dampers in Deaerators ė.

Hot side (total quantities)

Cold side

2,240 ft<sup>2</sup> of 3-3/4" FN-90 MASPAC or equal (top layer) 2,240 ft<sup>2</sup> of 3-3/4" FN-90 MASPAC, or equal (bottom layer)

5,000 ft<sup>2</sup> of 3-3/4" FN-90 MASPAC or equal (top layer) 5,000 ft<sup>2</sup> of 2" FN-200 MASPAC or equal

(bottom layer)

87

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study Sheet 3 of 8

# 9.1 EQUIPMENT LIST FOR 250 MGD VTE PLANT (Cont'd.)

Evaporators and Exchangers (Cont'd.) Total No. Required In Plant

### f. Entrainment Separators

Vertical hook-and-vane type for horizontal gas flow. Installed along one side of each vertical and horizontal tube bundle.

g. Flashing Brine Trays : 4 (1 per train)

h. Product Water Trays : 8 (2 per train)

Pumps and Drivers

Pump Type	ertical dry it centrifu-	gal, synchronous motor.	olute, vert.	haft, bottom	uction centri	'ugal, induc-	ion motor.	Mixed flow,	ultistage,	wet-pit verti-	al, can-type	entrifugal,
	316ss V	1 <i>p</i> 0 ¤	316ss v	Ø	Ø	4		316ss M	E	A	ပ	O
Materials Case Impeller Shaft			31688					316ss				
Case	D-2 CF8M Ni-resist SS		D-2	Ni-resist				D-2	Ni-resist			
Motor HP	2250		100					3500				
RPM	327		7000					004				
Head	,99		12.2'					162,				
GPM Each	115,000		27,200					72,500				
Service	Sea Water Intake		Booster Pump					Make-up Pump				
Total No. Required In Plant	<b>†</b>		†7					7				

synchronous

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study

Sheet 4 of 8			Pump Type	Mixed flow, single- stage, vertical wet-pit can-type centrifugal, induction motor	Mixed flow, multistage wet-pit, can-type vertical centrifugal, synchronous motor	) )Mixed flow, single-	stage, vertical, wet- bit centrifugal pumps	for all eff motors are i	All shafts on effect pumps employ seals to prevent air in-leakage.			Horizontal double-suction centrifugal, induction motor	Horizontal, centrifugal "canned-pump type", induction motor.	- ditto -	- ditto -	- ditto -	- ditto -
			Shaft	31688	31688	Monel	Monel	Monel	31688	31688							
	rs (Cont'd.)	Materials	Case Impeller	D-2 31688 Ni-resist	D-2 316SS Ni-resist	D-2 D-2 Ni-resist 316SS	Ni-resist 316SS	Ni-resist 316SS	Ni-resist 316SS	Ni-resist CF8M		Monel	Steel	Steel	Steel	Steel	Steel
	Pumps and Drivers (Cont'd.)	Motor		i 350 -in	1500 -iN	500 II.	400 Ni-	350 Ni-	350 Ni-	50 Ni-		300	3/4	3/4	3/4	3/4	3/4
			RPM	009	514	514	009	720	720	860		860	1750	1750	1750	1750	1750
(Cont'd.)			Head	,04	109'	50.	50,	50.	50,	45,	spares.	65 psig	25 psig	15 psig	15 psig	25 psig	25 psig
		GPM	Each	27,900	44,200	35,000	27,500	22,400	22,400	3,200	spare or	5,200	5.1	2.2	2.0	3.4	4.0
EQUIPMENT LIST FOR 250 MGD VTE PLANT			Service	Blowdown Pump	Product Water	No. I Effect Pump	No. II Effect Pump	Nos. III through X Effect Pump	Nos. XI through XV Effect Pump	Sump Punp	*Includes warehouse s	Chlorine Injection Pump.	Acid Injection Pump	Caustic Injection Pump	Anti-foam Injection Pump	Ferrous Sulfate Injection Pump	Sodium Sulfite Pump
9.1 EQUIPMEN	Total No.	Required	In Plant	<i>†</i>	<i>\tau</i>	*	*	34* 89	* [2]	α		* **	* * &	* * *	***	***	**

\*\*Includes installed spares.

Sheet 5 of

## EQUIPMENT LIST FOR 250 MGD VTE PLANT (Cont'd.)

### Air Compressors and Drivers

(1/3 chargeable to water plant - 2/3 to power plant)

Required

rotal No.

In Plant

\* \* U \* \* \*

Non-lubricated air compressors

Lubricated air compressors

2000 SCFW @ 100 psi: 400 HP

psi:

@ 100

2000 SCFM

### Ejector System, Including Condensers

Ejector No. 1, first stage, 20" suction, Hastelloy C construction

Ejector No. 2, second stage, 12" suction, Hastelloy C construction

Barometric inter-condenser No. 1 on suction side of Ejector No. 2, 316 SS

Ejewior No. 3, third stage, 8" suction, Hastelloy C construction, discharge to atmosphere.

Barometric inter-condenser No. 2 on suction side of Ejector No. 3, 316SS

Each of the above 3-stage Ejector sets evacuates the deaerator on one train. Note:

area High-ressure vent condenser, for vents from stages at above-atmospheric pressure; 110 ft<sup>2</sup> each condenser; tube material 70-30 Cu-Ni; tube sheet material 70-30 Cu-Ni clad steel.

Includes ins elled spryes.

9.1 EQUIFMENT LIST FOR 250 MGD VTE PLANT (Cont'd.)

	500,000 gals.	79,000 gals.	66,000 gals.	50,000 gals.	10,000 gals.	55-ton capacity							
	56' Diameter X 27' Deep	35' Diameter X 11' Deep	30' Diameter X 13' Deep	30' Diameter X 10' Deep	9' Diameter X 21' Long		14' Diameter X 48' Deep	8' Diameter X 45' Deep	12' Diameter X 48' Deep	11' Diameter X 37' Deep	9' Diameter X 37' Deep	8' Diameter X 37' Deep	8' Diameter X 45' Deep
Tanks and Sumps	Steel, Cone Roof	Steel, Cone Roof	Steel, Cone Roof	Steel, Cone Roof	Steel, Horizontal							o No. IX, incl.	No. XV, incl.
	Sulfuric Acid Tank	Sodium Sulfite Tank	Ferrous Sulfate Tank	Caustic Tank	Anti-foaming Solution Tank with Mixer	Chlorine Tank Cars, with automatic change-over equipment	Sumps, Make-up Pump	Sumps, Blowdown Pump	Sumps, Product Water	Sumps, Effect Pump No. I	Sumps, Effect Pump No. II	Sumps, Effect Pumps No. III to No. IX, incl.	Sumps, Effect Pumps No. X to No. XV, incl.
Total No. Required in Plant	Ţ	~	ı	1	ч	α	7†	7	7	4	†	28	24;

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study Sheet 7 of 8

9.1 EQUIPMENT LIST FOR 250 MGD VTE PLANT (Cont'd.)

Total No. Required

Electrical Components	13,800 V. to 2400 V.	13,800 V. to 480 V.	13,800 V. to 480 V.	480 V. to 208/120 V.	480 V. to 208/120 V.	2400 V., 350 MVA	480 V., 75,000 A.	208/120 V.	13,800 V., 1000 MVA	125 V., 100 Amp-Hrs.
	Transformers	Transformers	Transformers	Transformers	Transformers	2.4 KV Switchgear	480 V. Switchgear	Lighting Panels	Circuit Breakers	Control Batteries w/chargers
In Plant	i,	(1	Φ	∞	7	77	<b>∞</b>	ω 12	Ŋ	13

Outdoor, weatherproof Outdoor, weatherproof Outdoor, weatherproof Drawout type, air

50 KVA 25 KVA

10,000 KVA 750 KVA 3,000 KVA

MSF and VTE Comparative Study Sheet 8 of 8 OSW Contract 14-01-0001-1830 Fluor Contract 4334

## 9.1 EQUIPMENT LIST FOR 250 MGD VTE PLANT (Cont'd.)

	Total Duty:
Miscellaneous	
	Chlorine Vaporizers
Total No. Required In Plant	α

450 SCFM gas at 11" Hg vacuum. (225 SCFM each eductor)
Mat'l. of Construction: Titan Total Duty: Chlorine Gas Jet Eductors Ø

Titanium

4600 lb./hr. of chlorine vaporized (2300 lb./hr., each vaporizer)

Traveling Bar Racks, with drive

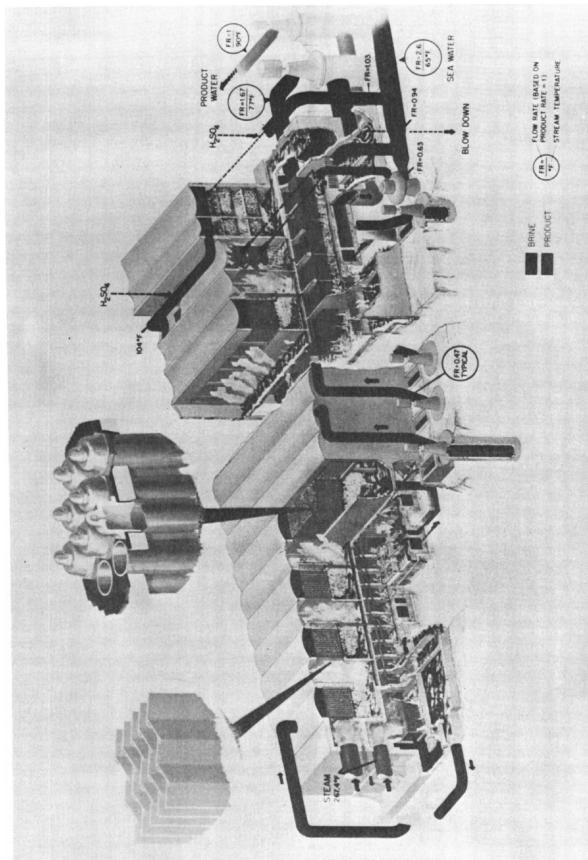
Traveling Screens, with drive

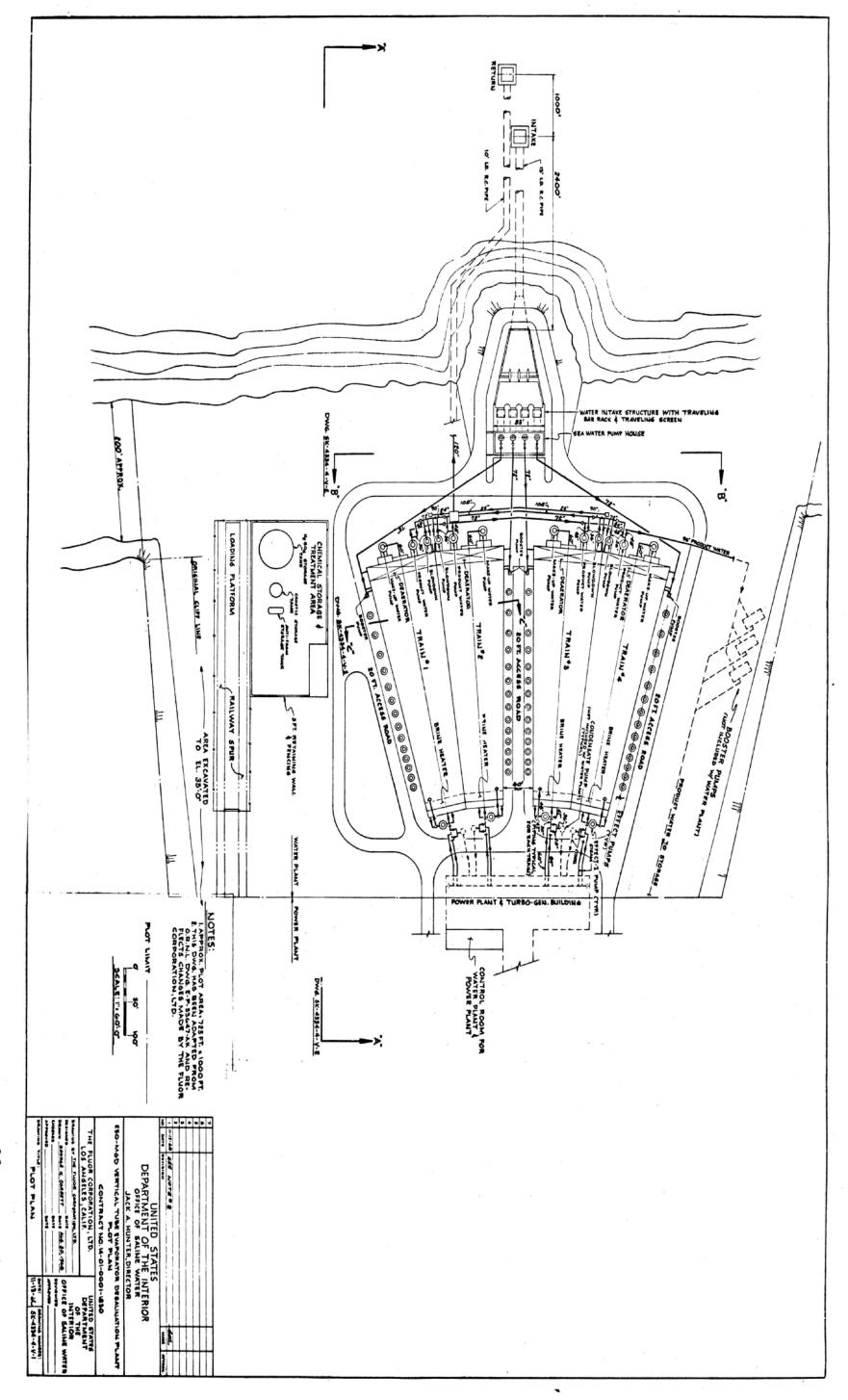
Crane, mounted over  $\boldsymbol{\mu}$  sea water pumps. (All other equipment is assumed to be serviced by mobile cranes.)

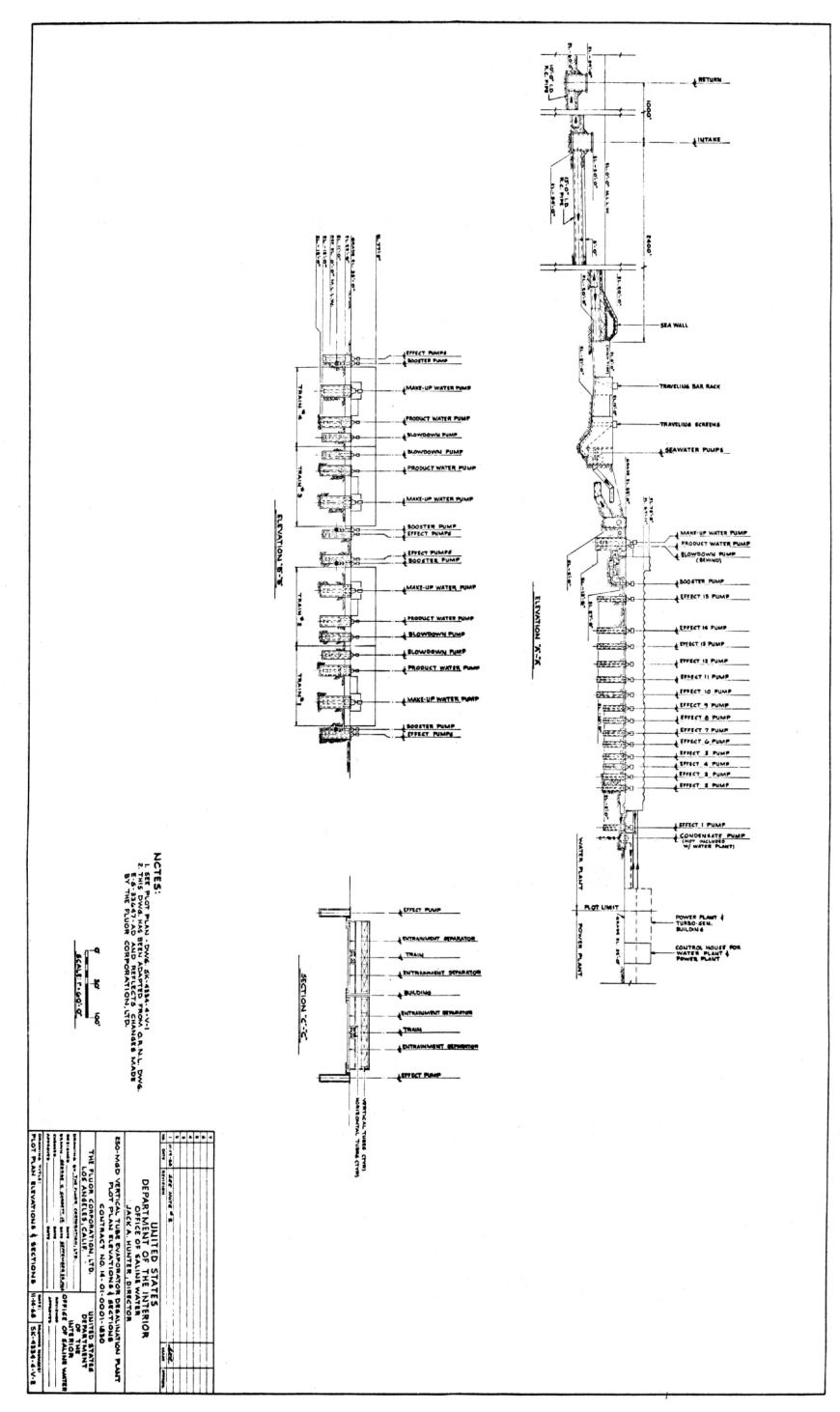
### 9.0 - 250 MGD VTE PLANT EQUIPMENT LIST AND DRAWING INDEX

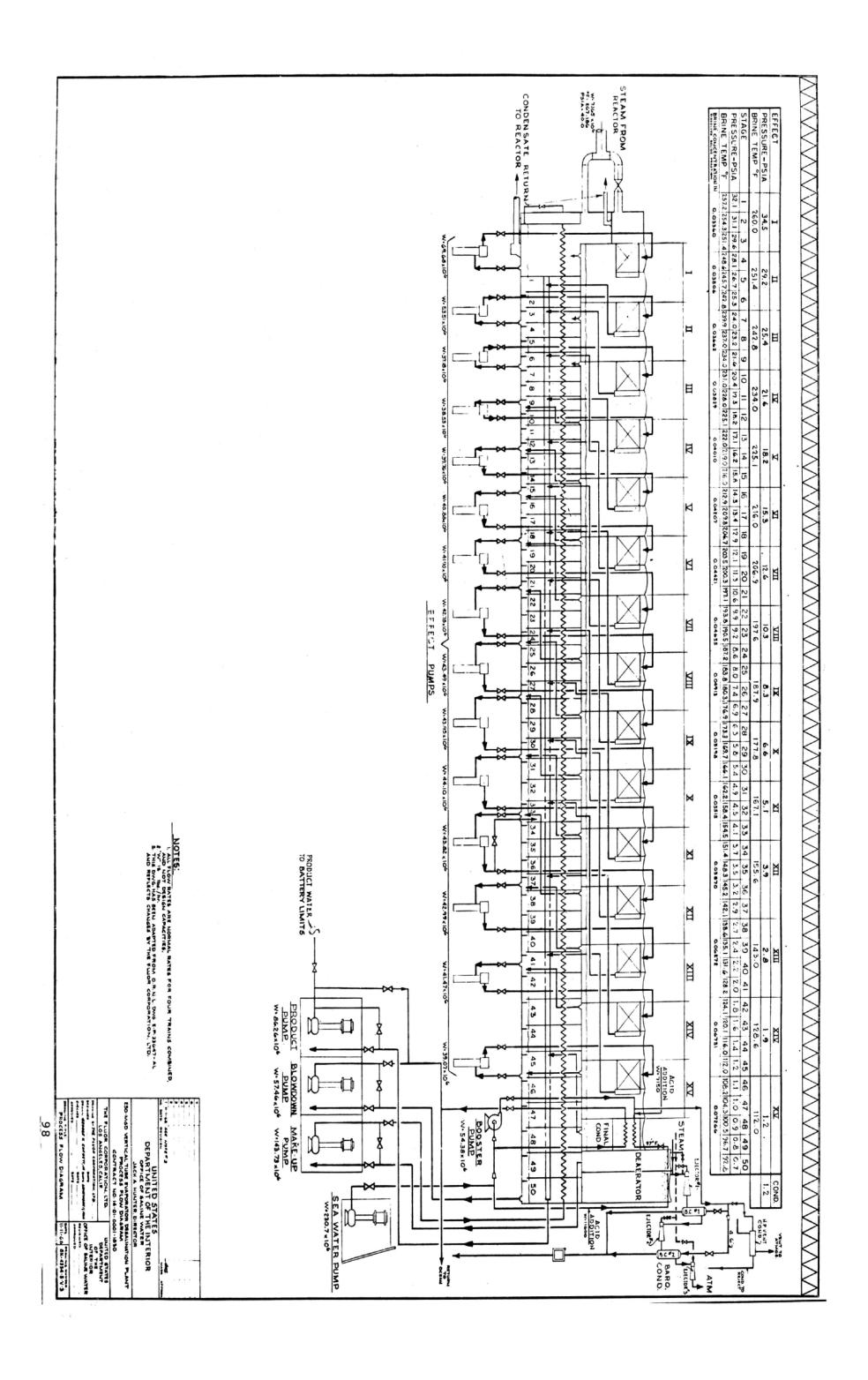
### 9.2 <u>Drawing Index</u>

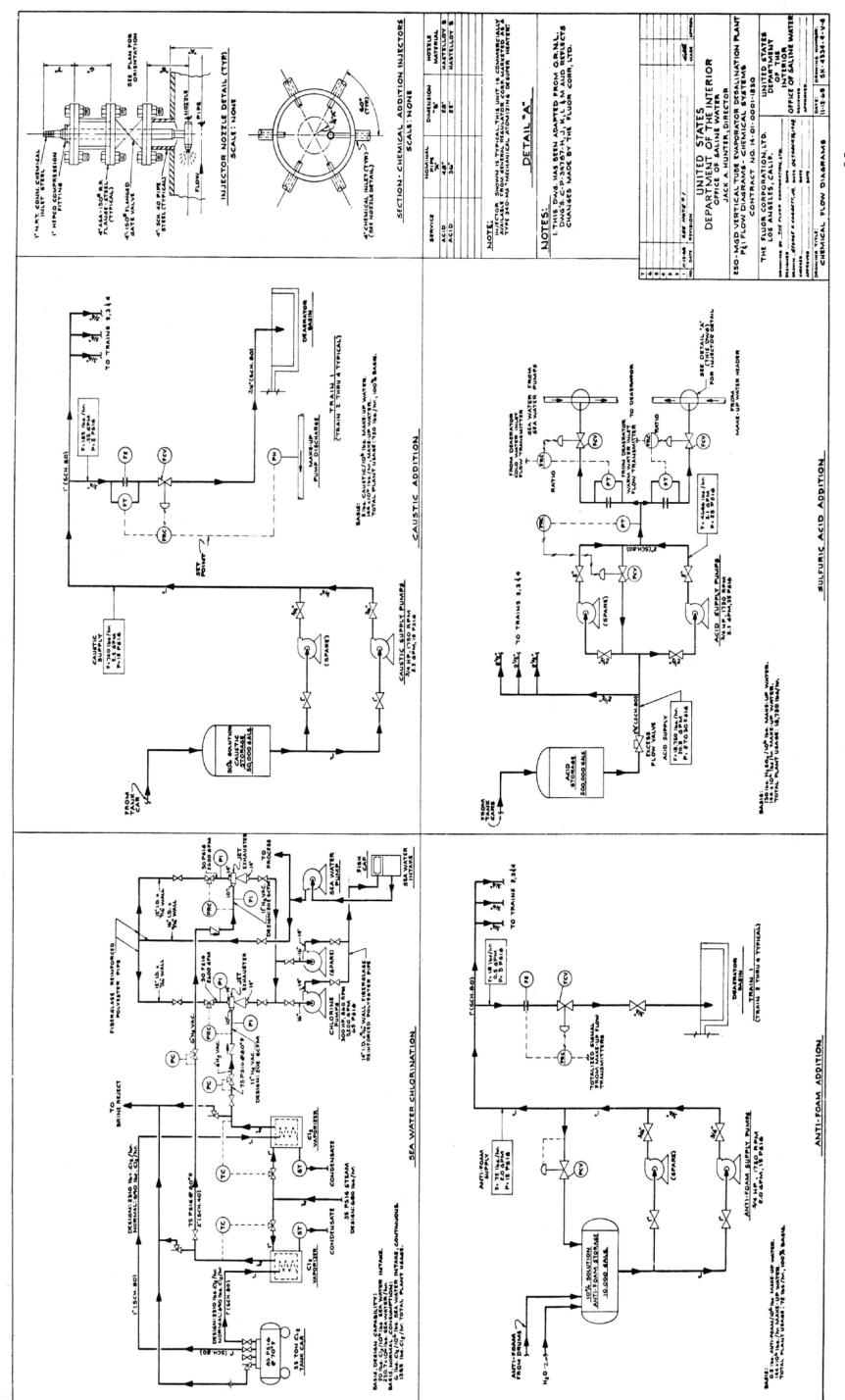
Drawing Number	<u>Title</u>
ORNL-A	Cutaway Drawing of VTE Plant
SK-4334-4-V-1	Plot Plan
<b>V-</b> 2	Plot Plan Elevations and Sections
<b>v-</b> 3	Process Flow Diagram
V-4	P&I Flow Diagrams - Chemical Systems
<b>v-</b> 5	One Train-Plan & Elevation
<b>v-</b> 6	Typical Effect - Plan & Section
V-7	Low Pressure End - Plan & Sections
<b>v-</b> 8	Cross Section & Tube Bundle Detail
<b>v-</b> 9	Longitudinal Section
V-10	Electrical One-Line Diagram

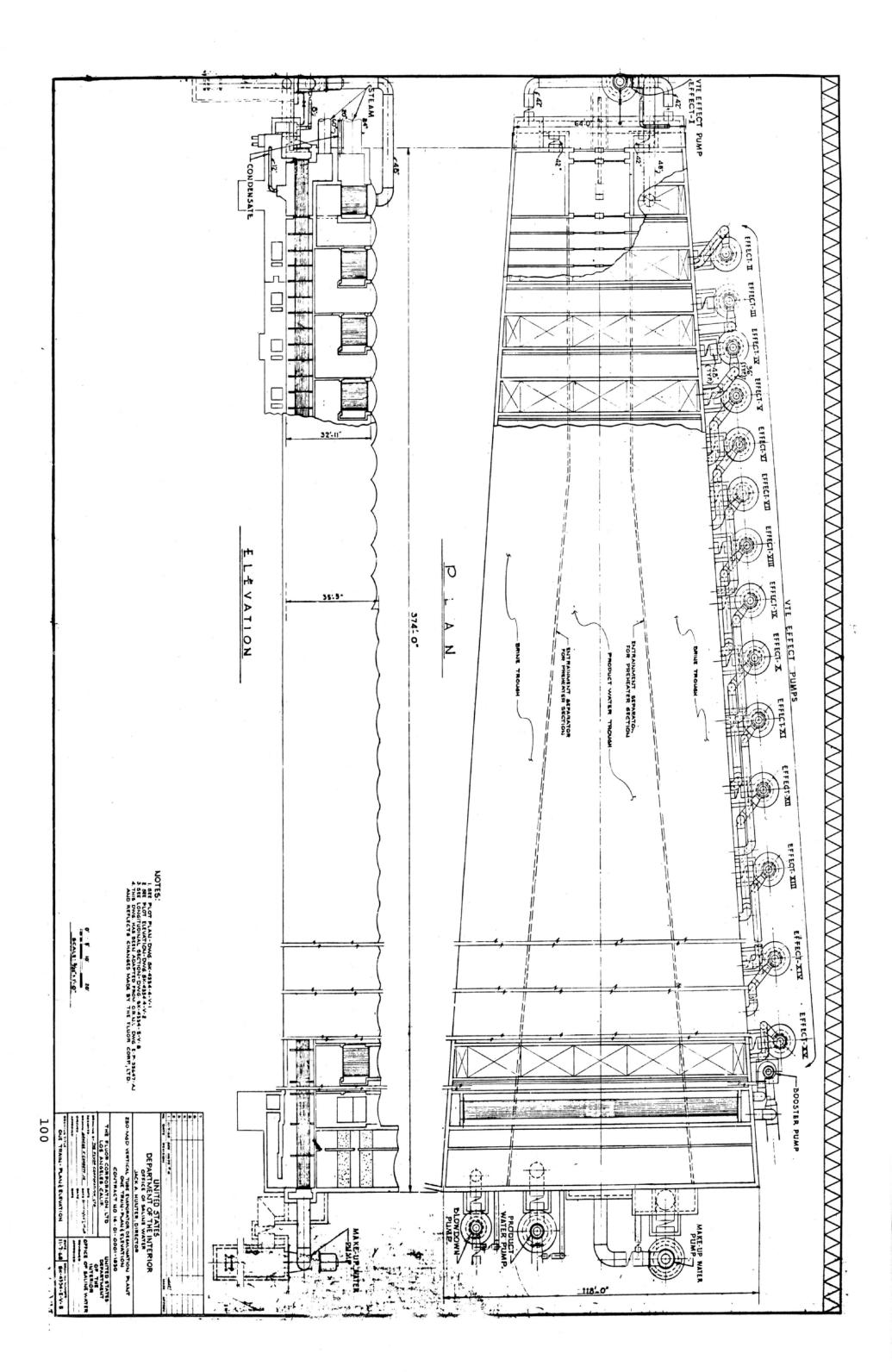


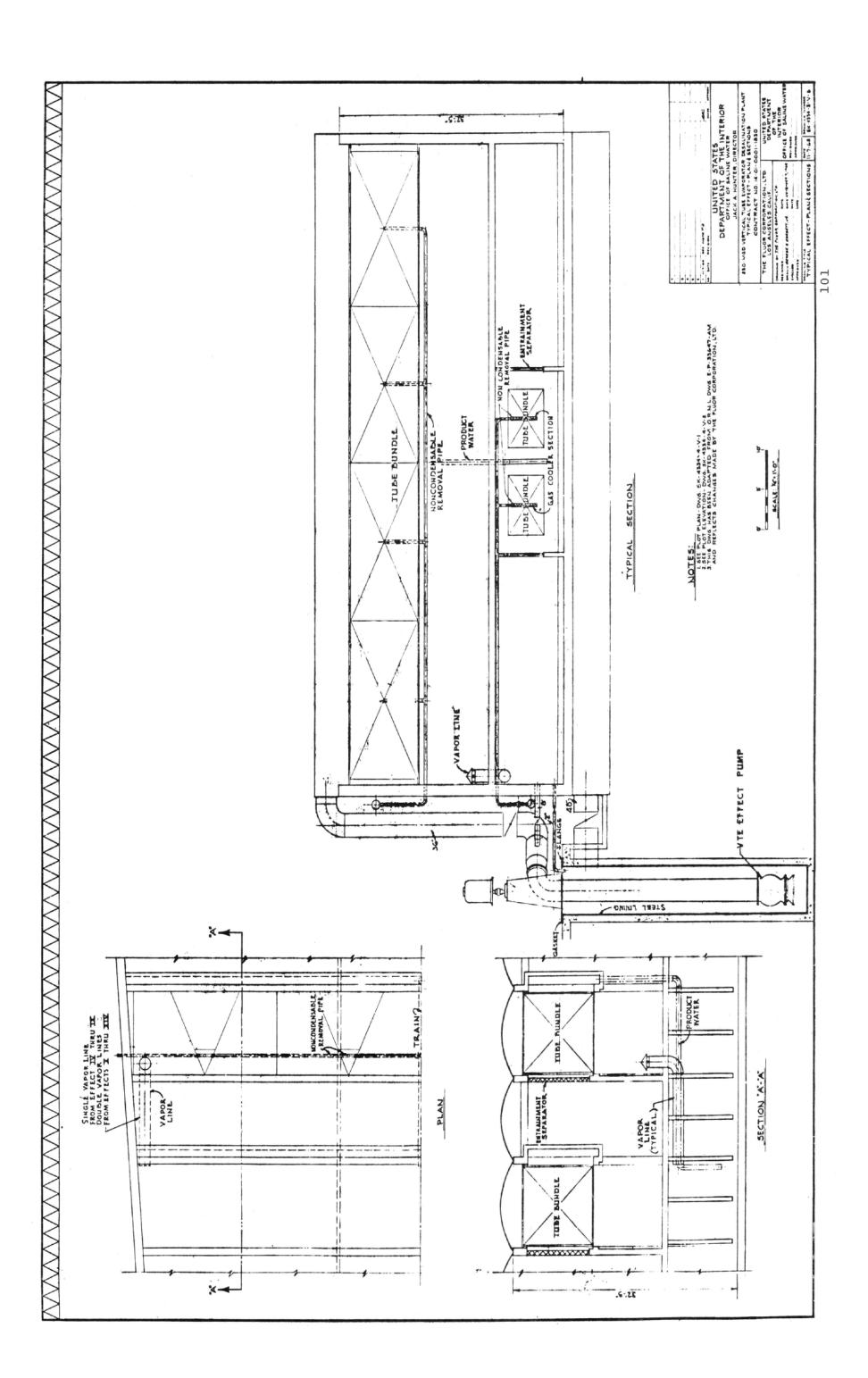


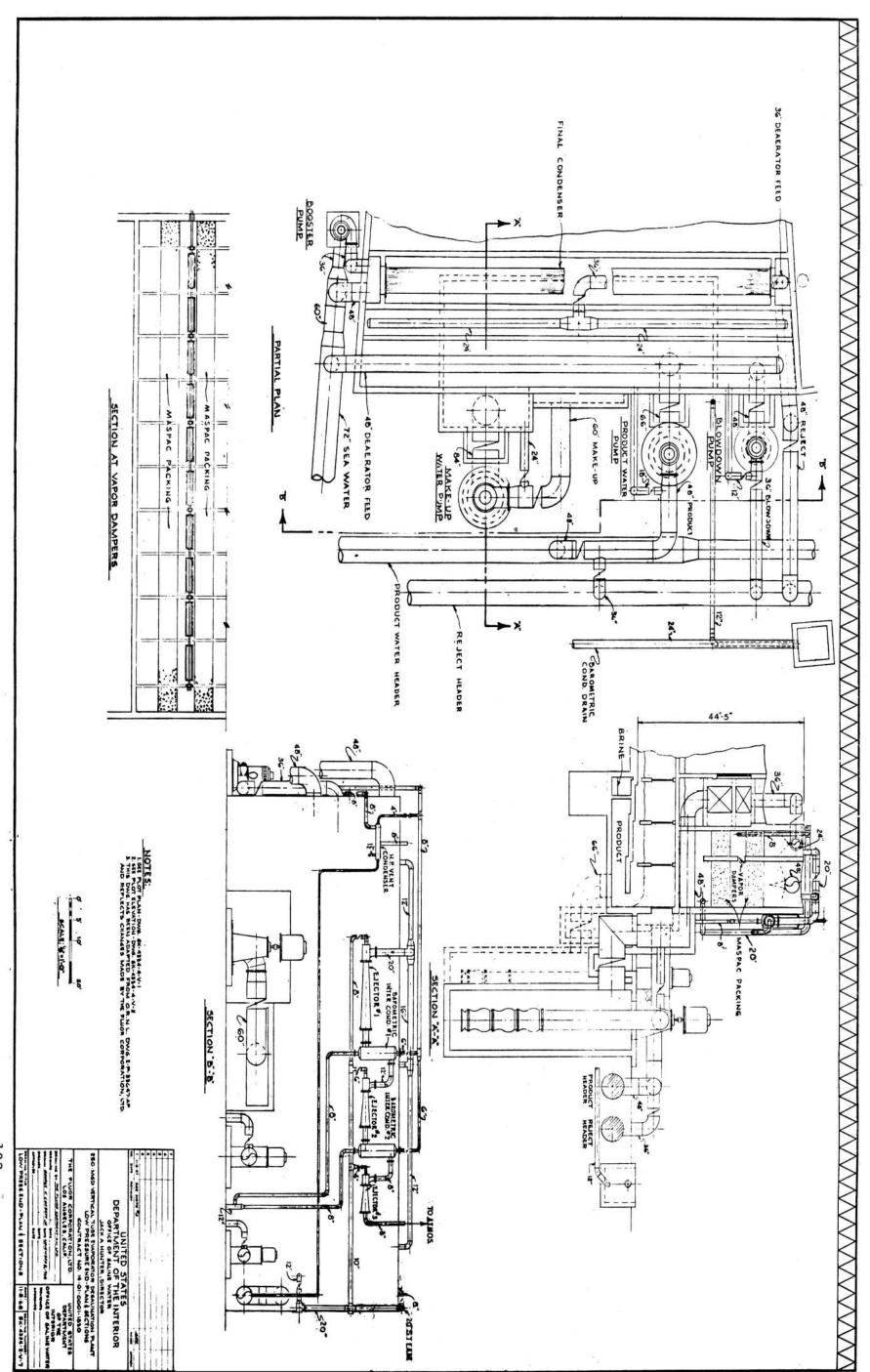


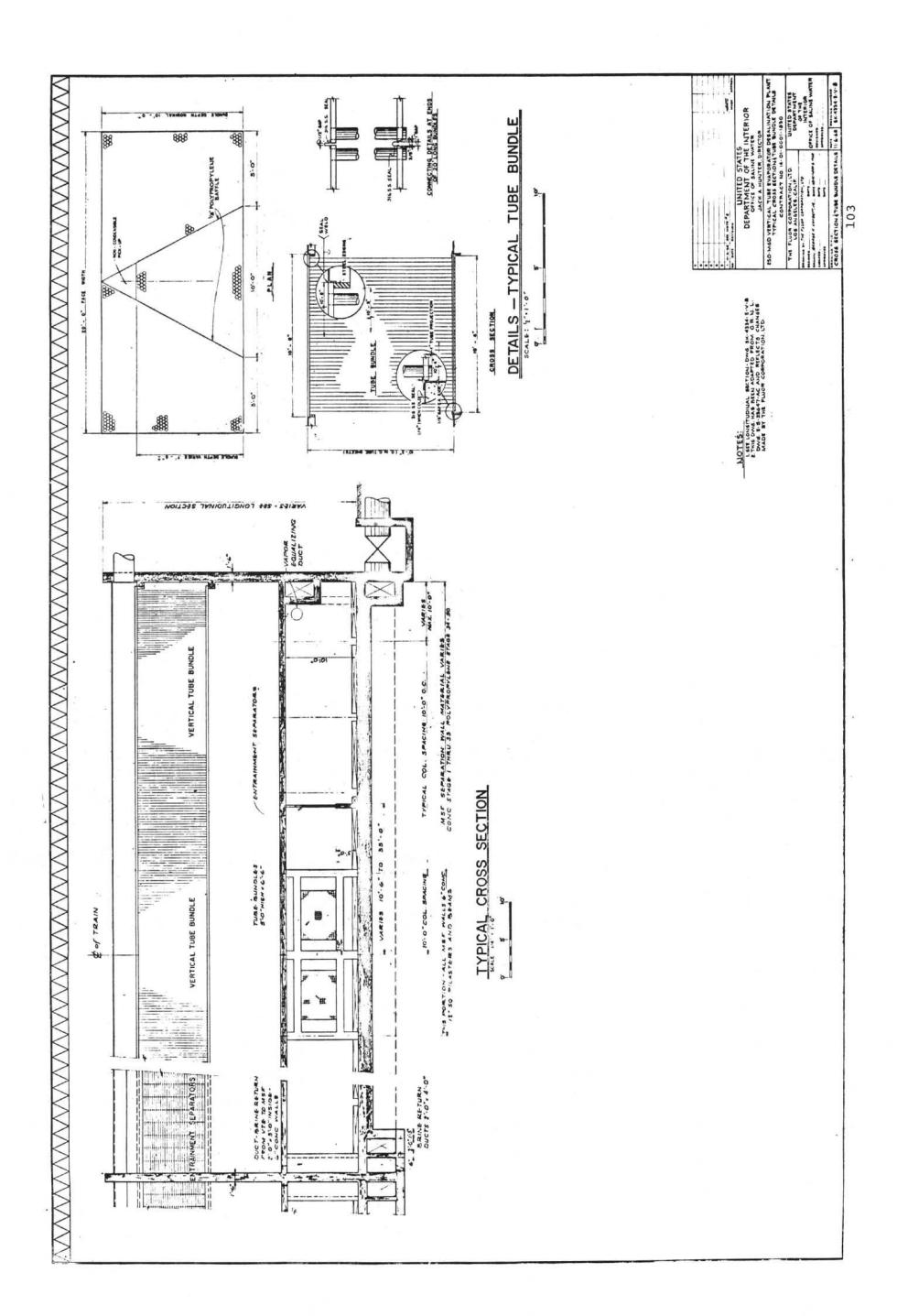


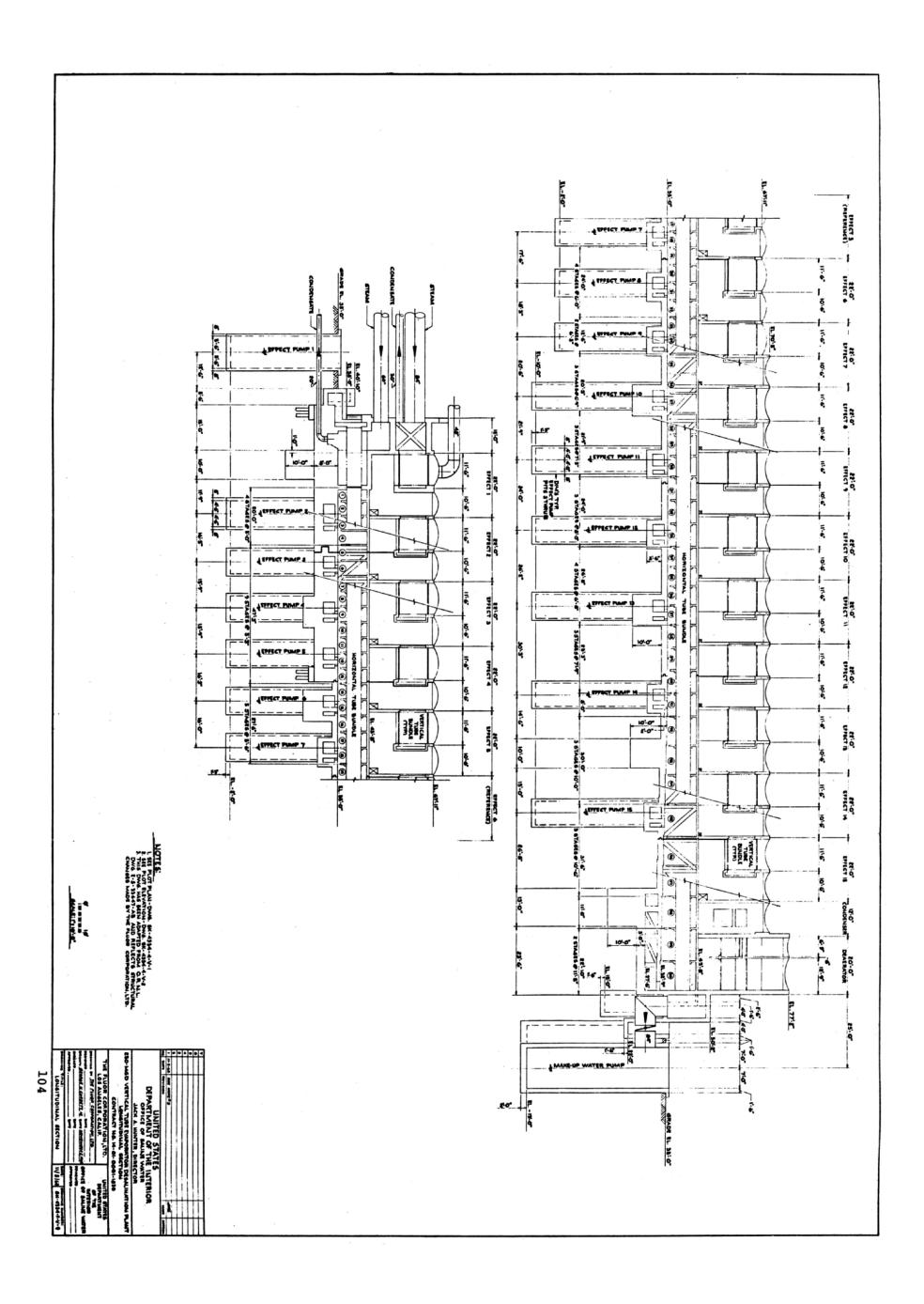


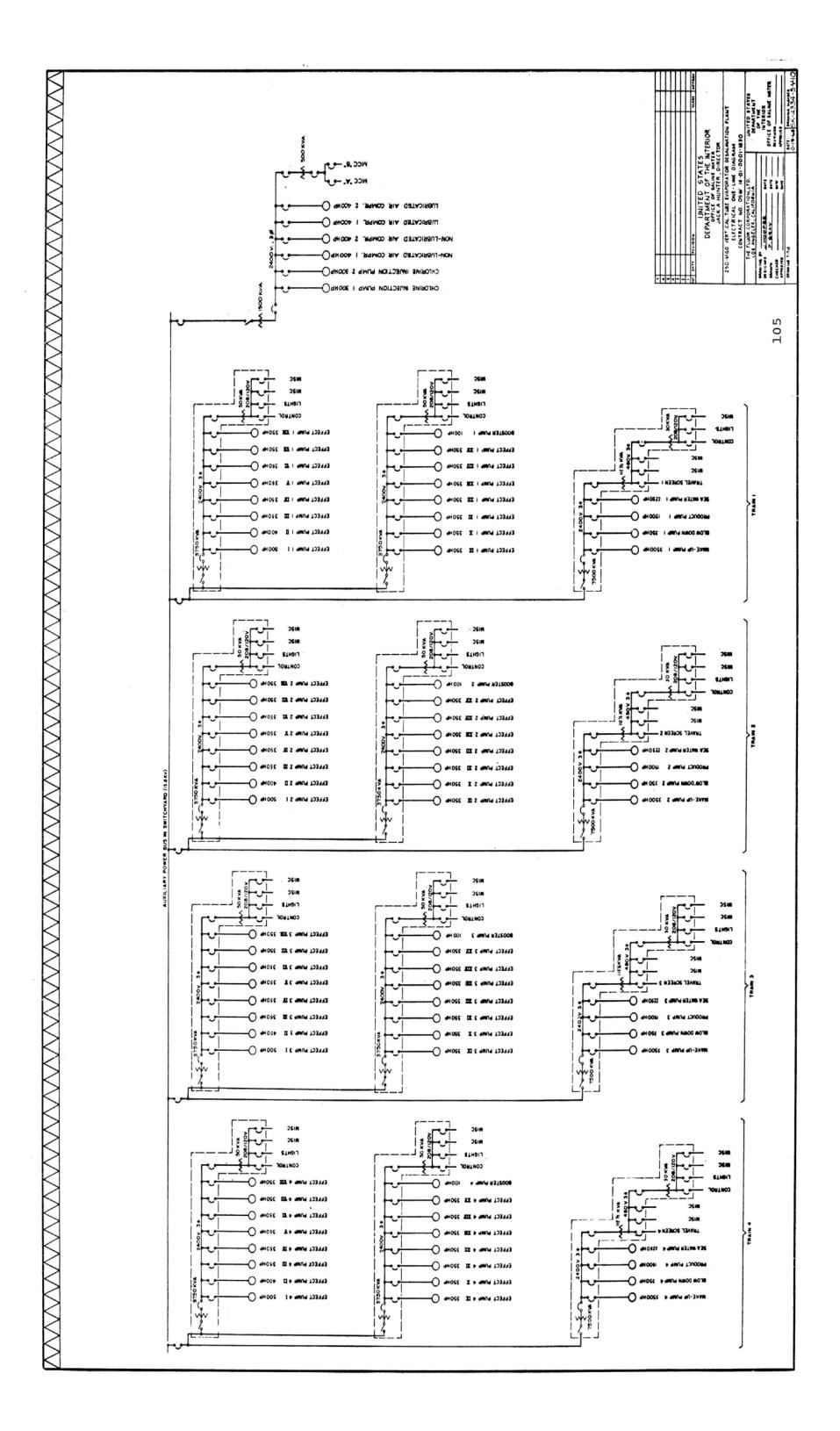












OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study Page 1 of 8

### 10.1 EQUIPMENT LIST

FOR

# 250-MGD MULTISTAGE FLASH (MSF) PLANT

Evaporators and Exchangers

REQUIRED IN PLANT

TOTAL NO.

77

Evaporator Vessels (Buildings) (Each contains 1 train)

Overall dimensions, each building:

692'L X 105'W X 24'H

Structural Steel

Construction, high-temperature end (approximately 152 ft.):

Construction, low-temperature end (approximately 540 ft.):

Concrete with steel liner

Each evaporator vessel contains several integral parts which will be separately itemized below. The quantities shown are for the complete 250 MGD plant unless otherwise noted.

(1) Heat Recovery Tubes (Stages 1-48)

Total surface area:

Total number of tubes:

Tube size:

Tube material:

Total number of tube bundles:

23,370,000 ft.<sup>2</sup>
247,960
3/4" 0.D. X 0.035" X 238.5'
and 241.5' long
90-10 cu-Ni

40 (10 per train; 5 in the low-temperature end and 5 in the high-temperature end)

(Cont'd.)
MSF PLANT
FOR 250 MGD
EQUIPMENT LIST FOR
ज्ञ १.०१

OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study Page 2 of 8

## Evaporators and Exchangers (cont'd.)

TOTAL NO. REQUIRED IN PLANT Heat Rejection Tubes (Stages 49-50) (2)

Total surface area:

Total number of tubes:

Total dimensions:

3/4" 0.D. X 0.049" X 67.5' long

1,352,000 ft.<sup>2</sup>

101,960

8 (2 levels per train)

20 (5 per train)

20 (5 per train)

90-10 Cu-Ni

Tube material:

Total number of tube bundles:

Flashing Brine Trays: (3)

Product Water Trays: (±) Wire Mesh Type Demisters (2)

Spray Deaerator 9)

Number of spray stages:

Number of spray nozzles:

Spray nozzle size:

1000 gpm at 5 psi delta p.

688

Saran

Spray nozzle material:

Brine Heaters  $(25^{1}-8\frac{1}{4}$ " 0.D. X 25'-5-3/4" tan to tan)

 $\infty$ 

Total surface area:

Total number of tubes:

Tubes:

Tube material:

1,187,000 ft.<sup>2</sup>

250,400 (31,700 each)

3/4" 0.D. X 0.035" wall X 24'-1-3/4" long

90-10 Cu-Ni

OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study Page 3 of 8		REWARKS	REMARKS  ( All pumps in these five services are vertical dry- ( pit type centrifugal pumps ( with top suction and side discharge, and are driven ( by synchronous motors.				Vertical wet-pit type centrifugal with bottom suction and side discharge, and induction motors.	Horizontal centrifugal with induction motor.	Horizontal centrifugal "canned pump type," with induction motor.	Horizontal centrifugal with induction motor.			
		SHAFT	316 88	316 SS	316 SS	316 SS	316 ss	316 SS	316 SS	Steel	Monel		
			MATERIALS	CFSM	CF8M	CF8M	316 ss	CF8M	CFBM	316 SS	Stee1	Monel	
	Drivers	CASE	316 SS Clad	316 SS Clad	316 SE Clad	Ductile Ni-resist Type D-2	316 ss Clad	Ni-resist	Ni-resist	Steel	Monel		
	Pumps and	MOTOR	3,000	17,000	. 002	1,500	500	50	10	3/4	450		
PLANT (Cont'd.)		RPM	327	225	225	514	1,50	860	1150	1750	1 860		
LANT (C		HEAD	571	189'	2hr	107	101	45.	20.	25 psi	65 psi		
10.1 EQUIPMENT LIST FOR 250 MGD MSF PI				GPM	152,000	310,000	96,600	002 <b>,</b> 44	42,300	3,200	1,500	6.1	6,800
		SERVICE	Sea Water Intake	Brine Recycle	Deaerator Feed	Product Water	Blowdown	Sump Pumps	Ejector Condenser Booster Pumps	Sulfuric Acid Injection	Chlorine Injection		
न ।.०।		TOTAL NO. REQUIRED IN PLANT	7	<b>ካ</b>	7	7	. <b>⊅</b> 108	α	4	*	* CU		

<sup>\* -</sup> Includes installed spares.

<u> </u>
(Cont'd.
MENT LIST FOR 250 MGD MSF PLANT (
GD MSF
MGD
250
FOR
LIST
EQUIPMENT LIST FOR 250 MGD MSF P
10.1

		Study	1
0SW-14-01-0001-1830	Fluor Contract 4334	MSF & VTE Comparative	Page 4 of 8

## Pumps and Drivers (Cont'd.)

		REMARKS	Horizontal centrifugal "canned-pump" type with induction motor	ditto	ditto	ditto
		SHAFT	Steel	Steel	Steel	Steel
	MATERIALS	IMPELLER	Steel	Steel	Steel	Steel
		CASE	Steel	Steel	Steel	Steel
	MOTOR	HP	3/4	3/4	3/4	3/4
		RPM	1750	1750	1750	1750 3/4
l		HEAD	2.4 15 psi 1750 3/4	25 psi 1750	4.0 25 psi	15 psi
	GPM	EACH	2.4	3.4	0.4	2.0
		SERVICE	Caustic Injection	Ferrous Sulfate	Sodium Sulfite	Anti-foam
TOTAL NO.	REQUIRED	IN FLANT	* «	* ~	*	*

# (1/3 to Water Plant - 2/3 to Power Plant)

TOTAL NO. REQUIRED IN PLANT

660 SCFM @ 100 psi; 125 HP	660 SCFM @ 100 psi; 125 HP
Non-lubricated compressors	Lubricated compressors
*	*

<sup>\* -</sup> Includes installed spares.

# ).1 EQUIPMENT LIST FOR 250 MGD MSF PLANT (Cont'd.)

OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study Page 5 of 8

# Ejector System, Including Condensers

No.	В	Į.
AL 1	JIRE	PLA
ľď	REQ	. N

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#

4

construction.
ပ
Hastelloy
suction,
6-inch
stage,
first
#1,
Ejector
4

Barometric condenser on suction side of Ejector #1, 316 SS construction.

Ejector #2, second stage, 4-inch suction, Hastelloy C construction.

Barometric condenser on suction side of Ejector #2, 316 SS construction.

Ejector #3, third stage, 3-inch suction, discharge to atmosphere, Hastelloy C construction.

Barometric condenser on suction side of Ejector #3, 316 SS construction.

Each of the above 3-stage ejector sets evacuates Stages 11-48 in one train.) (NOTE:

Ejector #4, first stage, 30-inch suction, Hastelloy C construction.

Ejector #5, second stage, 16-inch suction, Hastelloy C construction.

Barometric condenser on suction side of Ejector #5, 316 SS construction.

Ejector #6, third stage, 10-inch suction, discharge to atmosphere, Hastelloy C construction.

Barometric condenser on suction side of Ejector #6, 316 SS construction.

Each of the above 3-stage ejector sets evacuates the deaerator in one train.) (NOTE:

Ejector #7, hogging ejector, 10-inch suction, discharge to atmosphere, Hastelloy C construction.

4

Each of the above hogging ejectors evacuates one train during startup operations.) (NOTE:

# 10.1 EQUIPMENT LIST FOR 250 MGD MSF PLANT (Cont'd.)

OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study Page 6 of 8

### Tanks and Sumps

TOTAL NO. REQUIRED IN PLANT

Vent pump sump - 8' X 10' X 12' deep below pump

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PLANT
MGD MSF F
MGD
250
FOR
LIST
EQUIPMENT
10.1

OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study Page 7 of 8

### Electrical Components

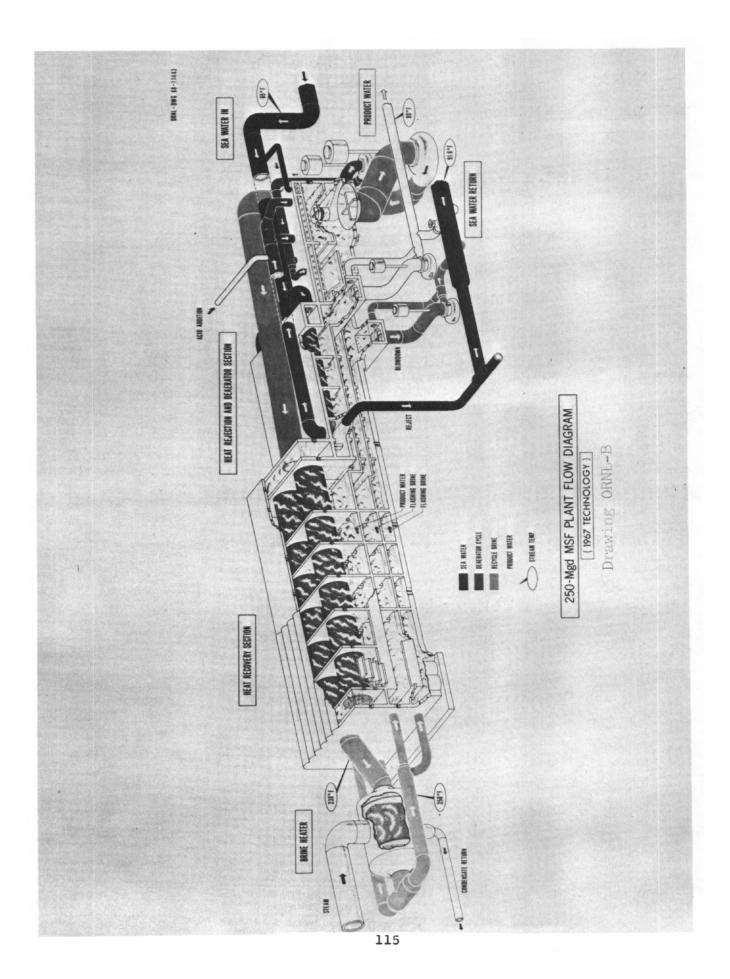
	13,800 V 2400 V.; 7500 KVA	2400 V 480 V.; 500 KVA	480 V 208/120 V.; 50 KVA	480 V 208/120 V.; 30 KVA	P. Motor	2400 V.; 350 MVA	480 V.; 22,000 Amp.	208/120 V.	13,800 V. Circuit Breakers - Drawout type - Air - 1200 Amp. 1000 MVA	125 V., 100 Amp-Hr
	Transformers:	Transformers:	Transformers:	Transformers:	Switchgear for 17,000 H.P. Motor	2,4 KV Switchgear	480 V. Switchgear	Lighting Panels	13,800 V. Circuit Break	Control Batteries w/Chargers
TOTAL NO. REQUIRED IN PLANT	†7	া ব	착	†	†7	†7	<b>⇒</b>	ω	7	V

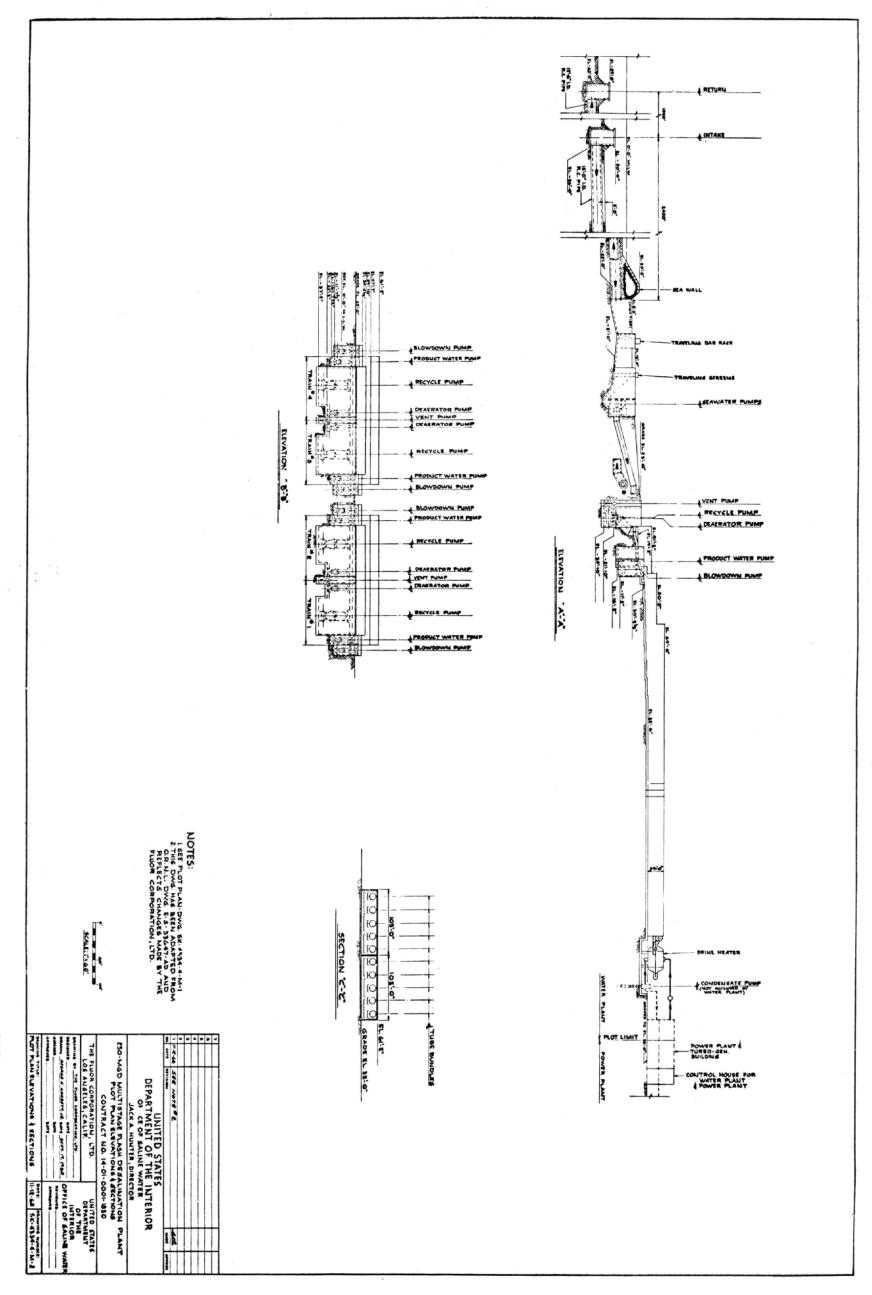
OSW-14-01-0001-1830 Fluor Contract 4334 MSF & VTE Comparative Study	rage o oi o		6,080 lbs. chlorine vaporized per hour (3,040 lbs./hr. each vaporizer) Material of Construction: Monel	600 SCFM gas at 11" Hg vacuum (300 SCFM each eductor) Material of Construction: Titanium			Crane, mounted over $4$ sea water pumps (All other equipment assumed to be serviced by mobile cranes.)
$\overline{}$	Miscellaneous		Total Duty:	Total Duty:			: pumps (All ot
EQUIPMENT LIST FOR 250 MGD MSF PLANT (Contid.)			Chlorine Vaporizers	Chlorine Gas Jet Eductors	Traveling Bar Racks w/Drive	Traveling Screens w/Drive	Crane, mounted over $4$ sea waten
10.1 EQUIP		TOTAL NO. REQUIRED IN PLANT	C)	α	<b>ಿ</b> 1∶	<i>⊒</i> 13	П

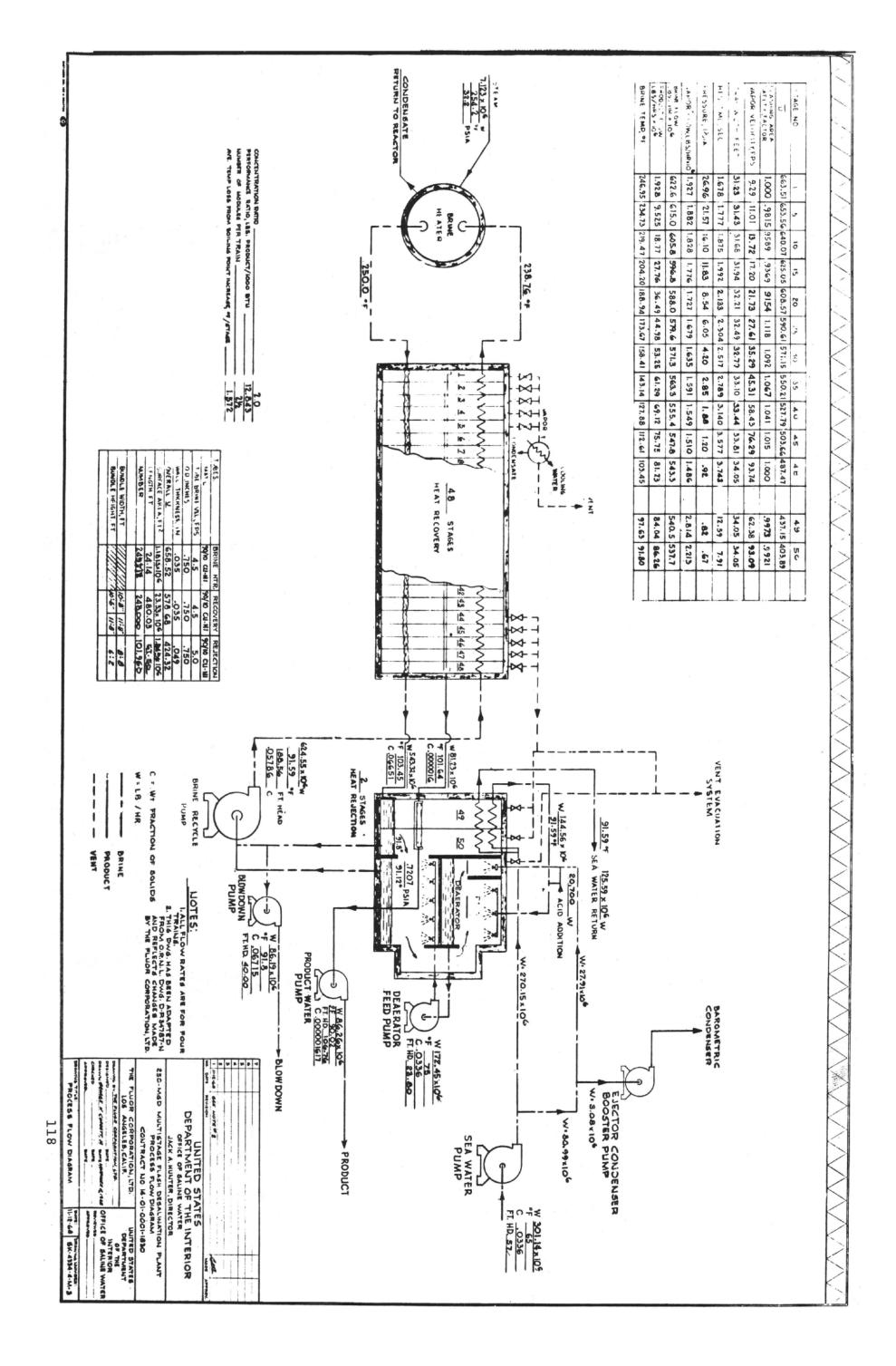
### 10.0 - 250 MGD MSF PLANT EQUIPMENT LIST AND DRAWING INDEX

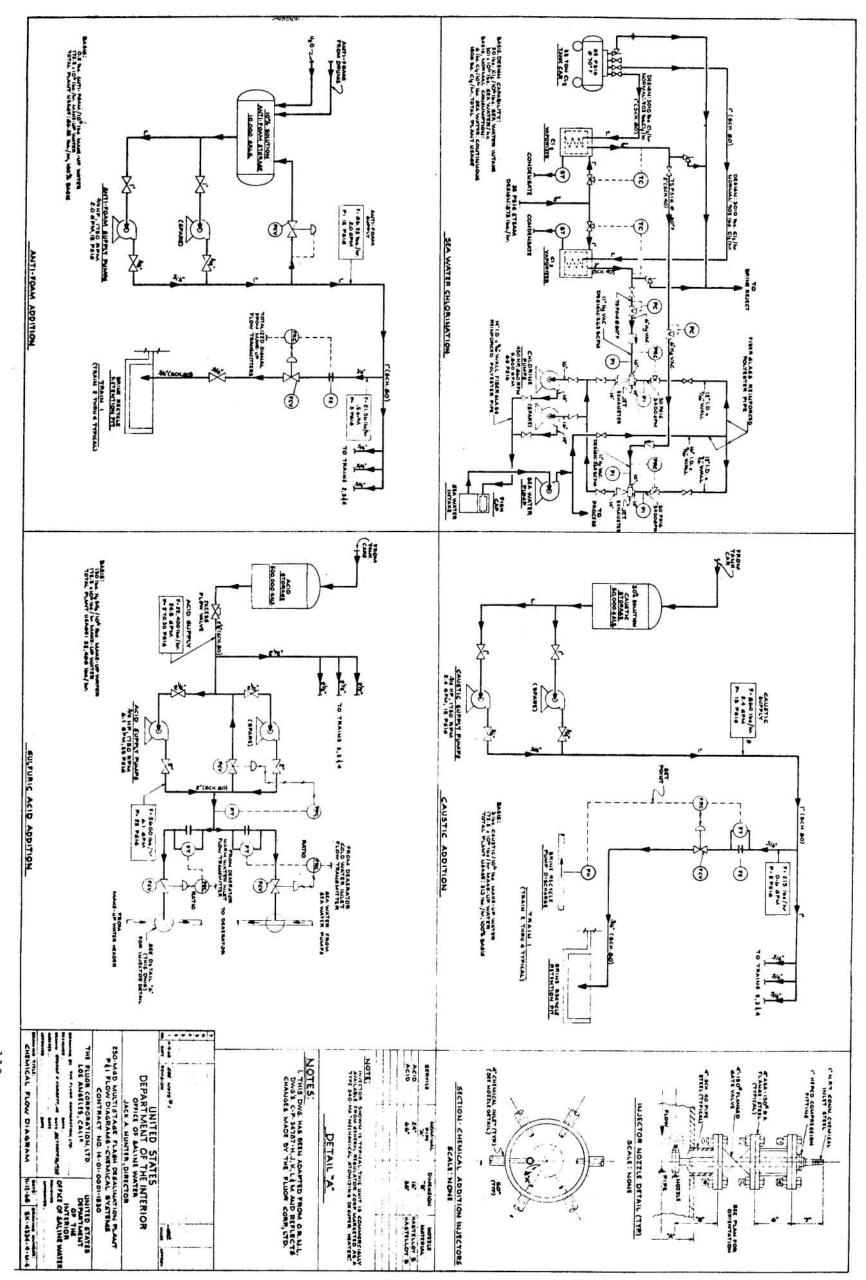
### 10.2 Drawing Index

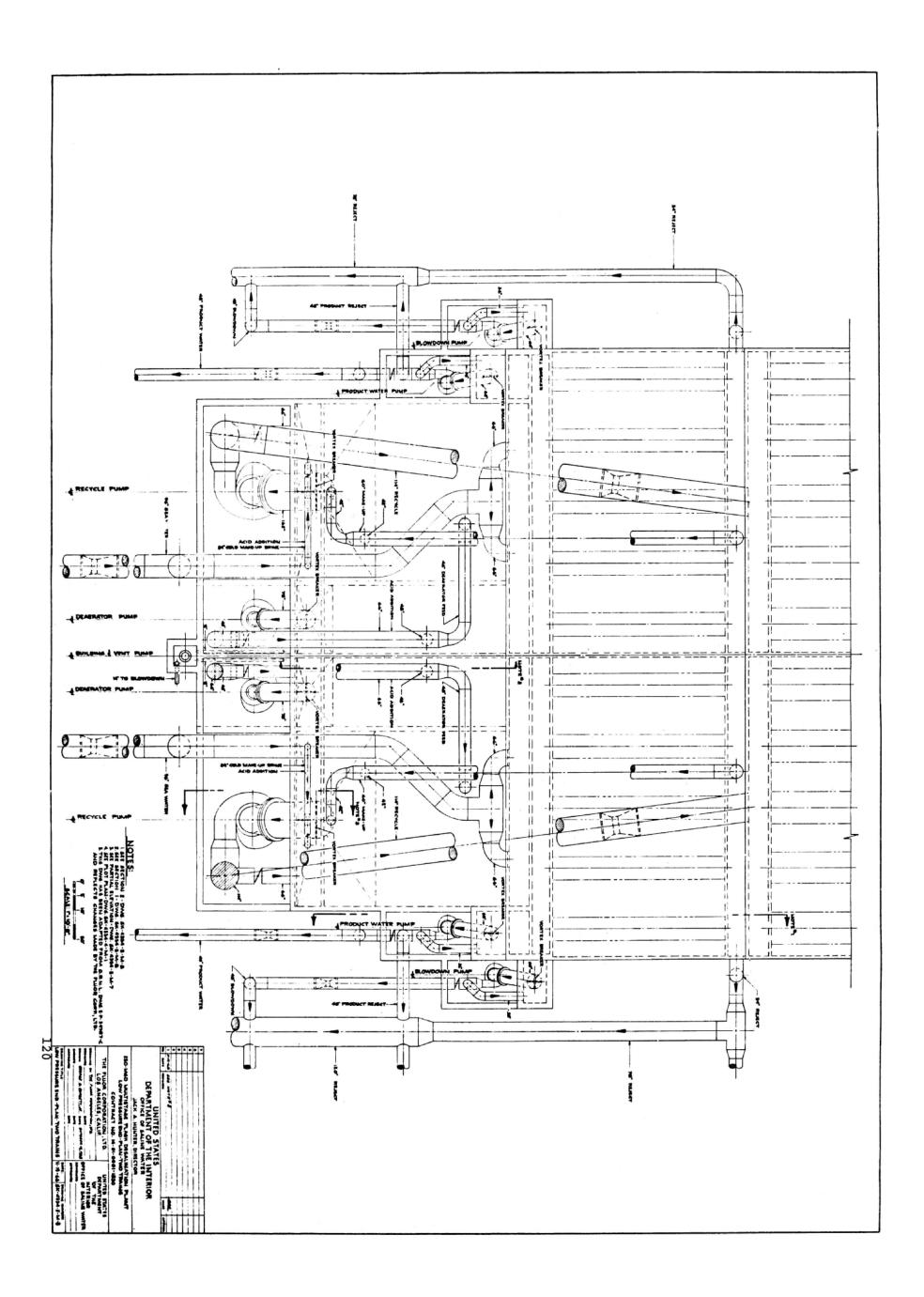
Drawing Number	$\underline{\mathtt{Title}}$			
ORNL-B	Cutaway Drawing of MSF Plant			
SK-4334-4-M-1	Plot Plan			
<b>M-</b> 2	Plot Plan Elevations & Section			
M-3	Process Flow Diagram			
M-4	P&I Flow Diagrams - Chemical Systems			
M-5	Low Pressure End - Plan, Two Trains			
M-6	High Pressure End - Plan, Two Trains			
M-7	Longitudinal Section			
M-8	Details			
<b>M-</b> 9	Electrical One-Line Diagram			

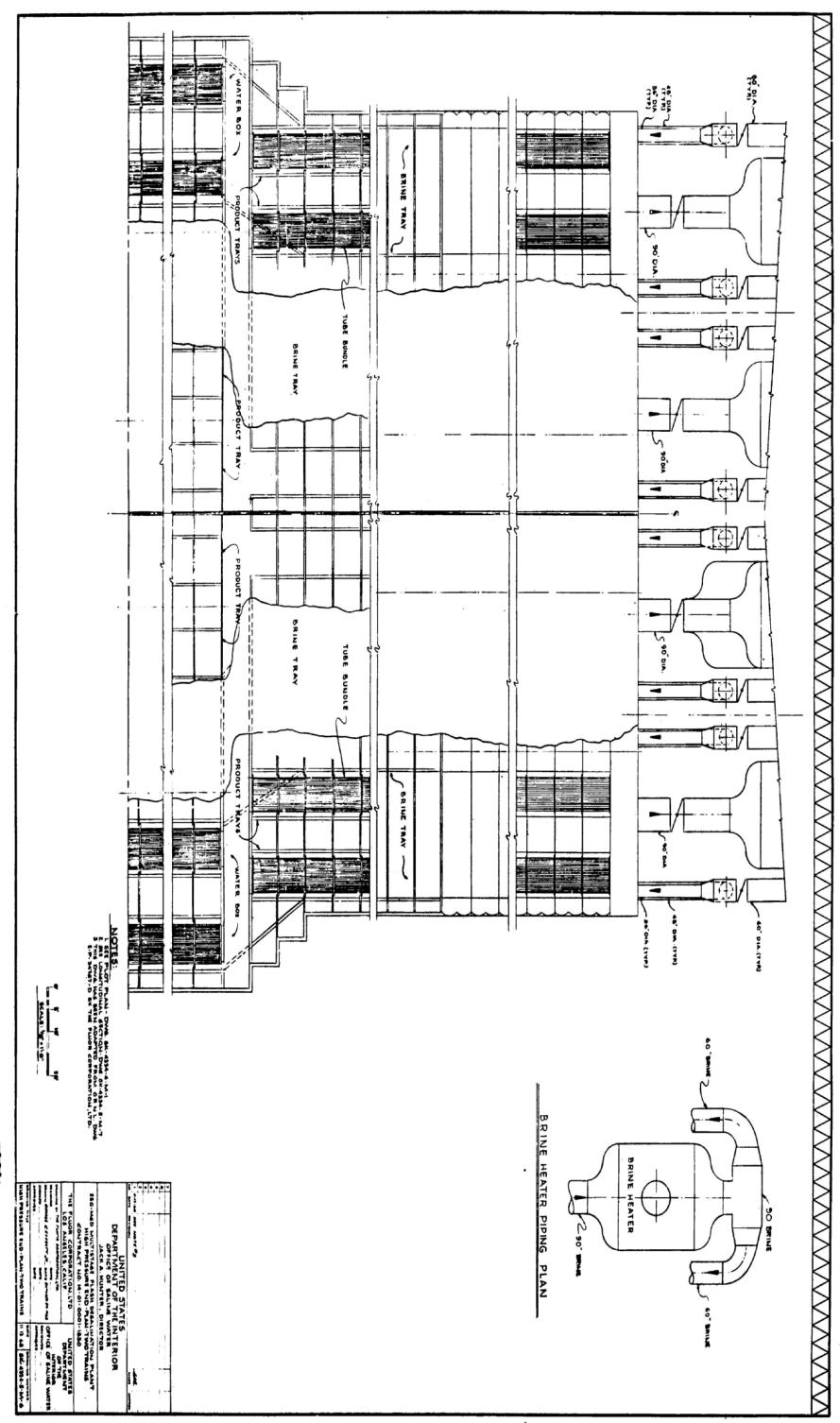


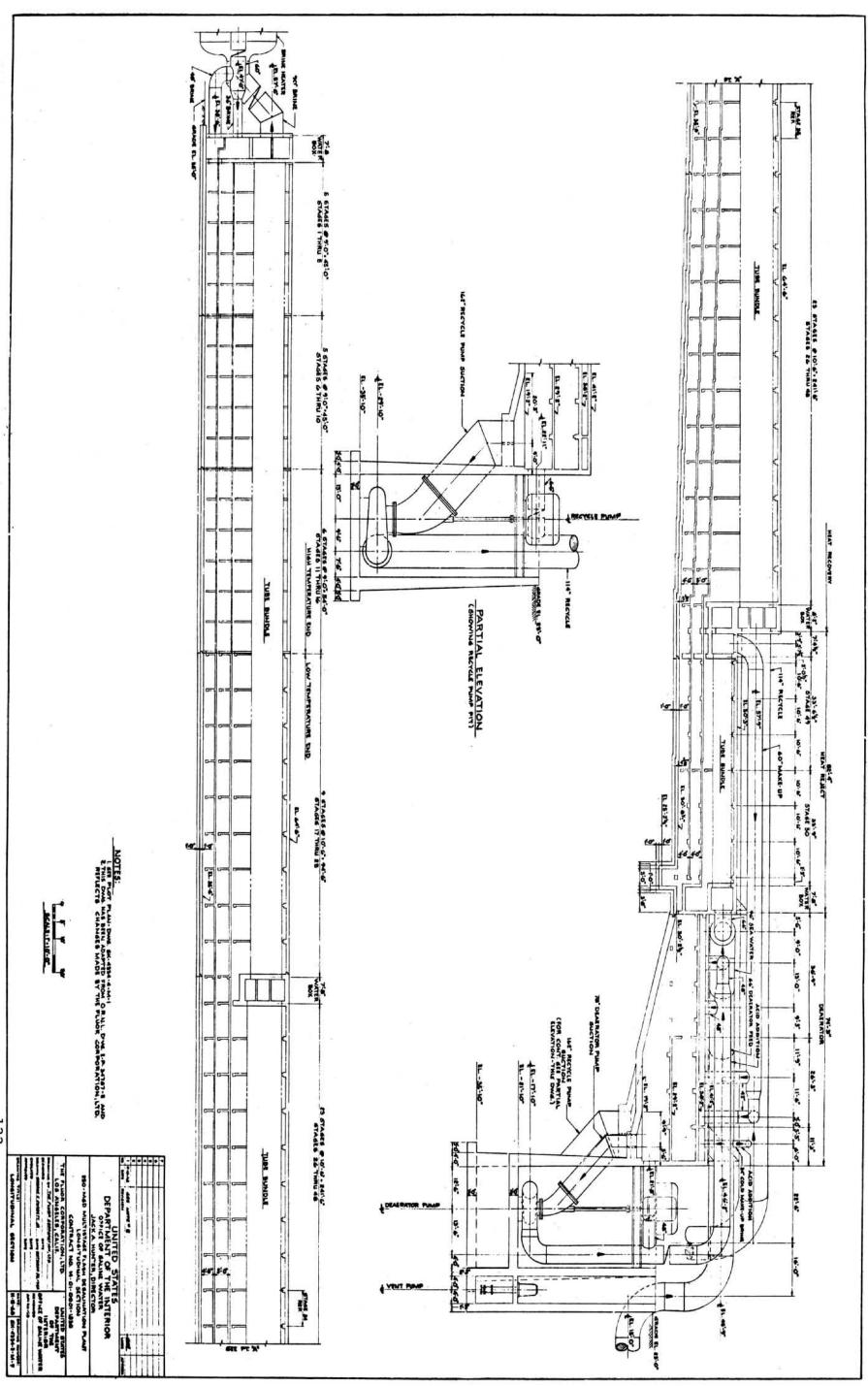


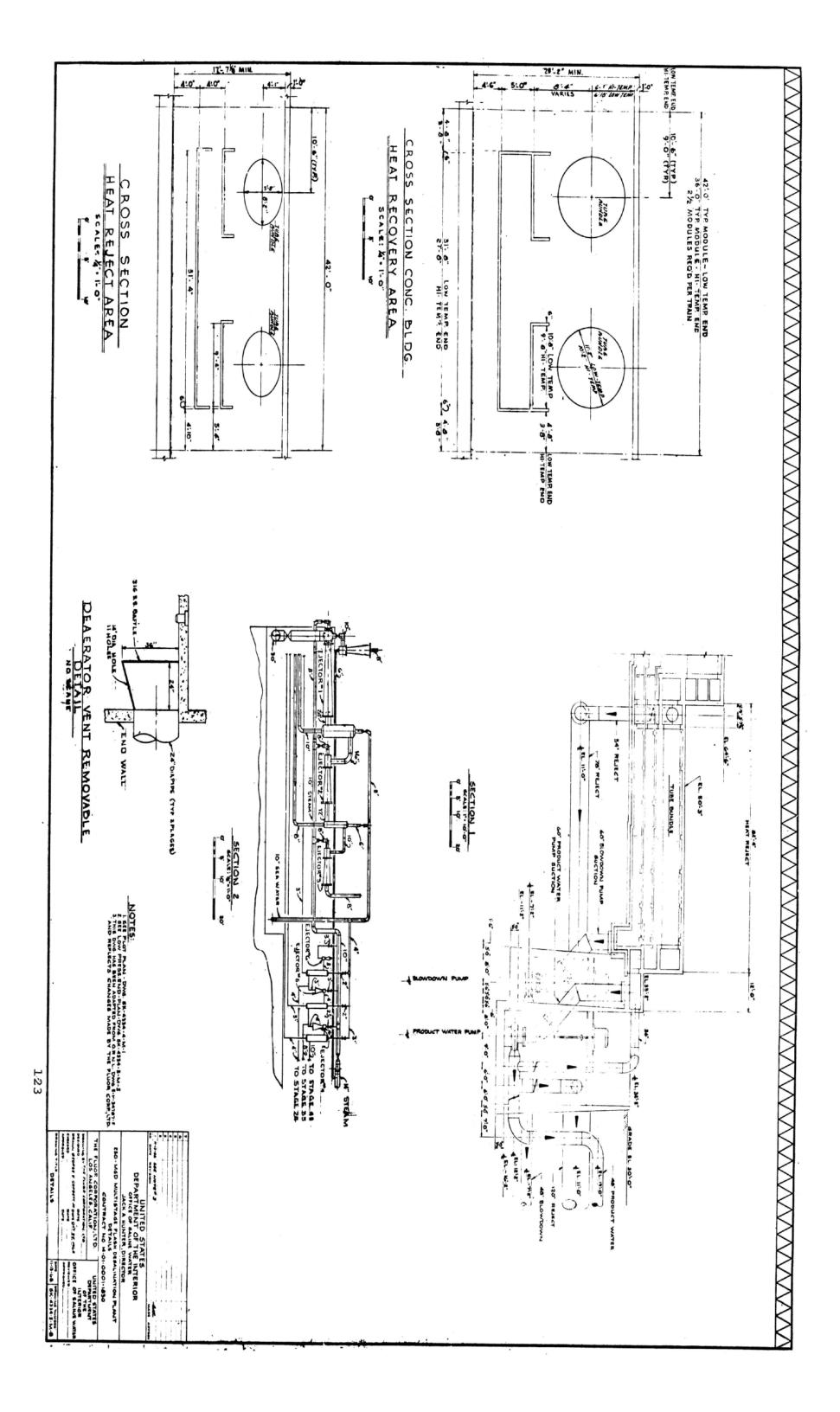


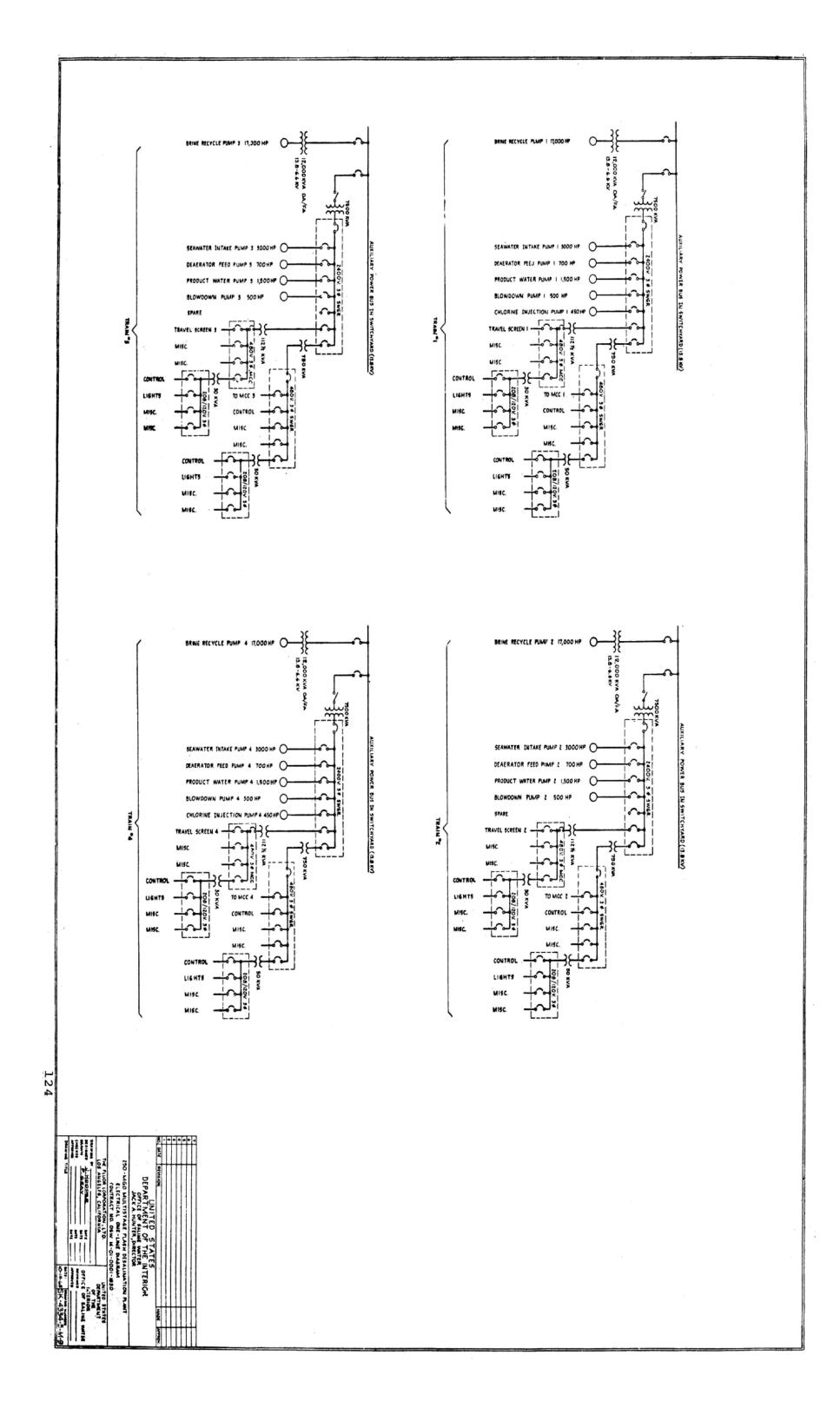












### 11.0 - TECHNICAL SUMMARY - 2.5 MGD PLANTS

### 11.1 VTE Plant Description - Process and Physical

### Process Description

The desalination process employed in the 2.5 MGD VTE plant is a falling-film, multiple-effect vertical tube evaporative process with 14 heat recovery effects. Accumulating condensate and seawater feed are flashed separately in a single flashing stage for each effect. The plant has a performance ratio of 11.6 pounds of product water for each 1000 Btu of input heating steam. The maximum brine temperature is 258 F. Other temperatures and flow rates of the process stream are shown in the Process Flow Diagram, Drawing 4-003.

Two plant configurations are evaluated in this study - one using conventional smooth tubes for all heat transfer, and one using enhanced-surface tubes. However, the heat and material balances are identical. (The reference source for the process and mechanical design of both plant configurations is given in Section 13.1. Enhanced-surface tubes are described in the latter portion of Section 11.1.)

Seawater is fed to the VTE plants on a once-through basis; thus the brine exposed to the highest evaporating temperature is only slightly concentrated. Seawater at 65 F is chlorinated at the outer end of the intake conduit to control biological growth, and screened to remove debris before being pumped to the plant. More than half of the seawater brought into the plant is used in the heat reject condenser to absorb the latent heat of condensation, and is immediately returned to the sea. Another 5% is diverted to cool the barometric condensers in the air ejector system, and the remainder becomes the seawater feed to the evaporator. The nondeaerated seawater feed pump forces the feed through the product water cooler, where the warm product heats it to 99 F., and then through the feed preheater in Effect 14, where the condensing vapor further heats it to 120 F. Sulfuric acid is added to neutralize its alkalinity, and the feed is then flashed into the combined deaeratordecarbonator, where nearly all of the dissolved oxygen, carbon dioxide, and other inerts are removed. The deaerated feed is then pumped through successive feed preheaters positioned in each effect, arriving at Effect 1 at a temperature of 251 F., where it becomes the feed to the first bundle of vertical falling-film tubes.

The VTE plant receives its heating steam from an outside source, and all condensate from the heating steam is returned to the source. The plant heating steam at 42 psia and 270 F is admitted to the shell-side of the Effect 1 falling-film bundle. The steam condenses on the tubes, releasing its heat of condensation to the preheated, deaerated seawater which is falling as a film on the interior surface of the tubes. This causes the seawater feed to boil violently, so that approximately 5% is vaporized as it falls through the vertical tubes. The mixture of seawater and newly-formed steam flows from the bottom of the heater tube bundle into the lower section of the Effect 1 evaporator, where the

seawater disengages from the steam and collects in the sump. The slightly-concentrated brine is then flashed forward to the sump of Effect 2, while the steam is directed to the shell side of the Effect 2 vertical tube bundle and feed preheater tube bundle. Knitted wire mesh entrainment separators are provided to remove entrained droplets of brine from the steam flow. In Effect 2, the vapor condensing on the outside of the vertical tubes performs the same function as the heating steam did in Effect 1, boiling more brine on the inside of the tubes, and producing additional steam to be passed on to Effect 3. This process continues in the same manner through the plant to Effect 14.

Feedwater flowing forward from the first effect brine sump to the second is exposed to the lower pressure found in the second effect, and part of it will flash to vapor. To provide the falling-film portion of the evaporation process in the second effect, brine is pumped from the Effect 2 sump to the top of the vertical tube bundle in Effect 2. There it forms a boiling liquid film which gravitates through the tubes and returns to the Effect 2 sump.

These essential processes are repeated in each effect through the plant, with each effect operating at a progressively lower temperature and pressure. The steam that is formed in the last (14th) effect is condensed at a temperature of 110 F. in the heat reject surface condenser. Seawater, circulating in the tubes, removes the latent heat and rejects it to the sea. The brine remaining in the sump of the final effect is pumped to waste. Condensate from the heat reject condenser is combined with the condensate which has accumulated through Effects 2 through 14 to form the product stream, still at a temperature of 125 F. This sensible heat is recovered and the product water cooled to 79 F by passing it through the product cooler, where the released heat warms the incoming seawater feed to 99 F.

In all effects (except the first) the falling-film tubes are fed by a pump which withdraws brine from the brine sump and feeds it to the top of the vertical bundle. Special distributors (slotted circular weirs) distribute the brine to the tubes, through which it returns by gravity to the sump. The brine is driven forward through the plant by the decreasing pressure gradient, with the flow rate controlled in most cases by properly sized orifices between effects. Improved control over the forward flow of brine through the plant is accomplished by pumping the brine forward to Effects 5, 9, 12, 13 and 14. In each of these cases the pump output is regulated by a level controller in the suction sump. In the effects in which the feed is pumped forward, flashing occurs in a small chamber adjacent to the top of the vertical tube bundle. The feed then overflows to the top of the vertical tube bundle and is distributed to the tubes. Since the pumped feed flow rate is greater than the optimum circulation rate through the vertical falling-film tubes, an overflow weir and return line is provided to bypass excess brine down to the brine tray in the same effect.

Continuous venting of all effects is necessary to remove noncondensable gases. Effects above atmospheric pressure are vented directly to the atmosphere through suitable throttling valves. Effects operating at subatmospheric pressures are vented through barometric condensers to steam-jet air ejectors.

### Physical Plant

The 2.5 MGD Vertical Tube Evaporator plant consists of a series of 14 effects, all but the first of which are contained in a single modular vessel. (Effect 1, to which the plant heating steam is supplied, is in a separate vessel.) Two design variations are considered - one using smooth heat-transfer tubes, and one using tubes with enhanced surfaces. Drawing SR-A shows a perspective view of the 2.5 MGD VTE plant, and Drawing SR-B is a cutaway view showing the internal construction of the module. These drawing are specific for the VTE plant with enhanced tubes; however, the smooth-tube plant is similar except for tube construction and somewhat different vessel proportions.

The modular evaporator vessel containing Effects 2 through 14 has a rectangular cross-section with semi-circular bottom. Approximate outside dimensions of the module for both conventional and enhanced plants are 14 feet wide and 120 feet long. Vessel overall height is approximately 34 feet for the conventional plant and  $22\frac{1}{2}$  feet for the enhanced plant. Arrangement of the VTE plants on the site is shown in Drawing 4-002 for the smooth tube case and Drawing 4-001 for the enhanced plant.

Effects in the principal evaporator vessel are separated from each other by bulkheads, with appropriate conduits provided to transfer vapor, brine and condensate from effect to effect without the necessity of external piping. Each effect contains a bundle of vertical falling-film tubes, and a bundle of feed water preheater tubes, as well as trays for containing liquid brine and condensate. The feed preheaters are two-pass tube bundles which in each effect heat the feed regeneratively inside the tubes while condensing water vapor on the outside. The falling-film tubes accommodate a recirculating flow of feed brine pumped from and returning to the brine sump in each effect. (For a complete description of all tubes in the VTE plants, refer to the Equipment List in Section 17.)

Feed preheaters in Effects 2 through 14 are shop-assembled and set into the evaporator module during its final assembly. The smooth-tube preheaters and heat reject condensers use smooth 3/4 inch tubes, 18 to 22 feet long; enhanced tube preheaters and heat reject condensers employ nominal 1-inch corrugated tubes 10 to 14 feet long. The smooth-tube product water cooler uses two exchangers, each containing 3/4-inch tubes 26 feet long; with enhanced tubes, a single exchanger is used containing 1-inch tubes 20 feet long. The product water cooler for both plants, and the Effect 14 feet preheater for the smooth tube plant only, are located outside the modular vessel; all other preheater bundles and the heat rejection bundle are built into the vessel.

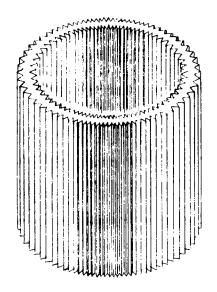
The seawater feed is raised nearly to its final temperature in the last preheater bundle, in Effect 2. From there it flows to the vertical falling-film tubes in Effect 1, which is a separate vessel. Heating steam is admitted to the outside of the tubes, evaporating part of the seawater feed and constituting the heat input to the plant. Details of the plant arrangement are found in the Mechanical Flow Diagram, Drawing 4-004, and process details in Drawing 4-003. Drawing 4-005 is a Chemical Flow Diagram, showing the control and addition of chemical additives, and Drawing 4-006 is an Electrical One-Line Diagram for the VTE plants.

As was mentioned earlier, two designs of VTE plants are reviewed in this study. In the first design, all heat-transfer elements are conventional smooth-surface tubes; in the second, enhanced-surface tubes are provided throughout - tubes whose walls are corrugated or fluted to improve heat-transfer rates. A brief discussion of enhanced tubes is appropriate at this point.

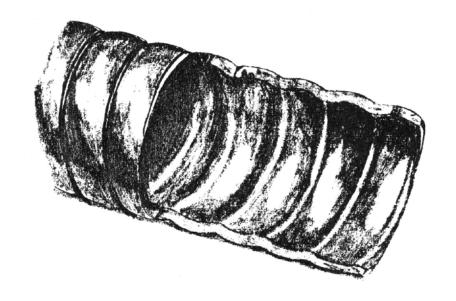
Although many variations exist, for the purposes of this study two general types of tube enhancement are recognized - one for vertical falling-film tubes and one for liquid-filled tubes. For falling-film tubes, the enhancement is double-fluting, with configuration similar to the tube section shown in Figure 11.1.1. The grooves or flutes represented in this sketch are about 1/8 inch apart, crest to crest, with moderately sharp tops and bottoms. The fluting is parallel to the tube axis, or may spiral slightly, but the spiral angle is always small. In vertical tube operation, heat is transferred from the condensing vapor on the outside of the tubes to the boiling liquid film on the inside. Surface tension causes the gravitating liquid on both sides to collect in the grooves, leaving the flute crests exposed.

For liquid-filled tubes, enhancement takes the form of corrugations similar to those shown in Figure 11.1.2. These corrugations generate turbulence in the flowing liquid inside the tube, and thus promote heat transfer from the boundary layer to the bulk fluid. The outside of the tubes may be in vapor, as with the feed preheater and heat rejection condenser tubes, or in liquid, as in the product water coolers. In contrast to the fluting in the vertical tubes, corrugations are oriented nearly at right angles to the direction of liquid flow, so that they present the maximum surface roughness to the flowing stream.

The overall designs of the VTE plants using smooth tubes and using enhanced tubes are identical except for the differing configurations of the tubes, and related changes. In general, the enhanced tube design allows the use of shorter tubes; consequently, the evaporator and exchanger vessels are also shorter. For example, the smooth falling-film tubes are 24 feet long, while their fluted counterparts are but 14 feet long. Another difference between the plants is in the pumping requirements. In the case of the corrugated tubes, the higher friction losses experienced by the tube-side liquid usually requires a moderate increase in pumping power.



### DOUBLE-FLUTED ENHANCED SURFACE TUBING



### CORRUGATED ENHANCED SURFACE

The proper distribution of the feed to the vertical falling-film tubes in each effect is accomplished by means of patented flow distributors which are inserted into the top of each individual tube. For the reference plant each flow distributor is a slotted circular weir so proportioned that a head of one to two inches of water is necessary to pass the required flow. This head requirement is sufficient to insure a reasonably uniform distribution of feed to all tubes across the tube bundle. The slots in the circular weir are so shaped that they inpart a tangential swirl to the feed flow entering each tube, thus helping to establish a uniform distribution of the falling-film over the inner surfaces of the vertical tubes.

Noncondensable gases are removed from each effect by appropriate vents. Effects operating at pressures below atmospheric are vented to steam-driven two-stage air ejectors. A barometric precondenser and intercondenser reduces the load of condensable vapors to the ejectors. A complete spare system of ejectors and condensers is installed to insure continuity of operation.

The deaerator and decarbonator are combined into a single vertical vessel 13'-6" in diameter and 26 feet high, which operates at 26 inches Hg vacuum. (Basis for this size is discussed in the following Study Procedure Section.) Deaerated feed flows by gravity from the deaerator-decarbonator to the vertical can-type deaerated feed pump.

Spares are provided for all pumps to minimize the incidence of plant shutdown for pump repair. Installed spares are provided for the seawater intake, raw seawater feed, deaerated seawater feed, heat reject distillate, product, acid, caustic, chlorine, and condensate return. In addition, one installed spare is provided for each two falling-film feed pumps and a spare is shared between the Effect 14 feed pump and the blowdown pump.

The tubing for both the smooth and enhanced tubes is 90-10 copper-nickel. The smooth-tube plant utilizes only seamless tubing; in the enhanced plant, welded tubes are used for the double-fluted tubes, and seamless for the corrugated enhanced-surface tubes. Tube sheets and channels of the preheaters and the heat reject condenser are solid copper-nickel or copper-nickel-clad over carbon steel base metal. All return heads are solid copper-nickel. In the falling-film tube bundles, the tube sheets are copper-nickel-clad over carbon steel, and water boxes are monel-clad or solid monel on the first five effects, and glass-rein-forced polyester on the balance.

The modular evaporator vessel consists of an externally reinforced carbon steel shell with double wall top section, designed so that it provides suitable steam passages and includes flash chambers for recovering heat from the product concentrate in each stage. All inner carbon steel plates in contact with seawater include a corrosion allowance in their design.

Knitted 316 stainless steel entrainment separators 4 inches thick are provided between the flashing liquids and the condensing tubes in the evaporator vessel. These separators are removable through the manways provided in each effect.

The combined deaerator-decarbonator is carbon steel with a shop-applied internal epoxy coating, and in addition, is provided with a 1/4 inch corrosion allowance. The seawater inlet nozzle is of 316 S.S., packing is polypropylene (Dow "Mas Pac") and the packing support plate is glass-reinforced epoxy.

The steam jet ejectors are built of 18-8 stainless steel, with the precondenser and intercondensers of rubber-lined steel or fiberglass.

In the reference design extensive use is made of fiberglass piping, including the following: seawater feed from the supply piping to the deaerator-decarbonator; heat reject cooling water; brine blowdown; deaerated feed from the deaerator through Effect 4; and effect pump piping from Effects 4 through 14. In the three highest temperature effects, 316 S.S. piping is used. The seawater intake line is of reinforced concrete. With minor exceptions, the remainder of the piping in the plant is of carbon steel.

The VTE evaporator is insulated for personnel safety and economy where fluid temperature exceeds 150 F. Evaporator insulation is of semirigid glass fiber panels fitted with aluminum covering, or of granules installed between the double walls of the evaporator vessel. Preformed or blocktype insulation is used on other vessel surfaces and on pipes. All except the granule-filled top section has 0.016 inch aluminum covering.

### 11.0 - TECHNICAL SUMMARY - 2.5 MGD PLANTS

### 11.2 MSF Plant Description - Process and Physical

### Process Description

The MSF desalination process described in this study of 2.5 MGD plants is a conventional multistage, single-effect flash evaporation process using recirculating brine flow. The reference source for the process and mechanical design evaluated in this portion of the study is given in Section 13.1. In this reference, the designer has provided a series of 11 process designs covering a spectrum of possible operating conditions, with the purpose of providing a universal plant design. Of the 11 designs given, 4 were for plants using polyphosphate treatment of the seawater feed, and were limited to a maximum brine temperature of 190 F. The remaining 7 plants, including the one chosen for this comparative study, used acid treatment of the feed, and were designed for a maximum brine temperature of 250 F. The basic plant configuration and module construction was similar for all of the 250 F plants, but the number of stages, operating temperatures and pressures in each stage, and performance ratio varied among the designs. Fluor has selected the Universal Plant Design No. 4 as being the one which most nearly matches the design conditions and performance ratio of the two small VTE plants. The plant contains 39 heat recovery stages and three heat rejection stages. Its performance ratio is 11.5 pounds of product for each 1000 Btu of heating steam, based on its nominal production capability of 2.5 million gallons per day. Temperatures, flow rates, and concentrations of the various process streams are shown in the Process Flow Diagram, Drawing 4-012.

Cold seawater at 65 F is pumped to the MSF plant by the seawater intake pumps as a combined feed and cooling stream. To control biological growth throughout the inlet system, it is chlorinated at the fish cap where the seawater enters the intake conduit. After screening to remove debris, a minor part of the seawater flow is diverted to cool the barometric condensers, while the major portion is pumped through the heat reject module to absorb the latent heat from the final flash stages. Nearly 75% of the warmed seawater leaving the heat reject module at 82 F is returned directly to the sea, while the remainder becomes the seawater make-up feed to the evaporator plant. Sulfuric acid is added to the feed to neutralize the alkalinity, and it is then sprayed into the combined deaerator-decarbonator. Under the joint influence of the very low pressure maintained in the deaerator and the stripping steam injected in countercurrent flow, the carbon dioxide, dissolved oxygen. and other inerts are removed to very low levels. If they are needed, caustic and antifoamer are added to the deaerated feed to adjust pH and prevent excessive foaming. The treated make-up is then added to the recycling brine by allowing it to flow by gravity into the suction of the recycle pump. (See Drawing 4-005 for the Chemical Flow Diagram.) The new seawater feed makes up 29% of the brine circulation stream, and the recycle brine from the last heat reject stage, which is twice as concentrated as natural seawater, constitutes the remaining 71%. control the recycle brine concentration, a portion of the brine stream from the last stage is diverted to blowdown before the new feed is added to the recycle stream.

The brine circulation stream is discharged from the recycle pump through the preheater tubes of each stage of the evaporator, acquiring heat from the condensing vapor on the way. Emerging from the evaporator, the circulating brine enters the brine heater, where it is heated by externally-supplied stream to its final temperature of 250 F, establishing its flashing potential. From there it flows through a pressure control valve to the flash chamber in the first stage of the evaporator, where a portion of the brine flashes to steam. The remainder flows through a flow-controlling orifice into the slightly lower pressure region of Stage 2, where additional flashing occurs. This process continues through the entire plant, with the brine flowing into regions of progressively lower temperatures and pressures in each successive stage. After leaving the final heat rejection stage, the remaining brine is at a temperature of 82 F.

The steam which flashes from the surface of the flowing brine stream in each evaporator stage passes through entrainment separators and is condensed on the outside of the preheater tubes through which the recycle brine is passing. This condensate, or product water, collects in a tray under the preheater tube bundles. Then, like the brine stream, it flows through flow-controlling orifices from stage to stage, dissipating its sensible heat by reflashing, until it finally emerges from the last heat-rejection stage into the suction of the product water pumps. Vapor flashed from the product stream recondenses on the preheater tubes, transferring its sensible heat to the recycle brine, and recovering the condensate. The temperature of the condensate stream has been reduced to 79 F when it leaves the final heat rejection stage; no product water cooler is necessary.

### 11.0 - TECHNICAL SUMMARY - 2.5 MGD PLANTS

### 11.2 MSF Plant Description - Process and Physical

### Physical Plant Description

The 2.5 MGD Universal MSF plant whose design is evaluated in this study is a modular plant assembled from a number of identical or near-identical components or modules. (The reference source of the process and mechanical design is given in Section 13.1). The typical recovery module in the MSF plant is a box-like structure approximately 90 feet long,  $8\frac{1}{2}$  feet wide, and 12 feet high, with flat sides and bottom and a domed top, and containing 8 evaporator stages. Four such modules connected together, plus a slightly shorter 7-stage module, plus a 3-stage heat rejection module, combine to form the 43-stage evaporator under comparison in this study. Drawings 4334-FC-A, FC-B, and FC-C show the high-temperature, low-temperature and heat rejection modules, respectively.

Modules are fabricated of carbon steel, using standard structural shapes and steel plate. The flat sides and bottom are reinforced with external steel ribs to resist the internal and external pressures. By keeping the overall width below 10'-0", the modules can be shop-assembled and transported to the site.

The arrangement of the units is shown in the Plot Plan, Drawing 4-011. Excluding the seawater intake and outlet structure, the entire water plant is placed on a plot 120' x 272'. The modules are connected together by appropriate piping to convey the recirculating brine feed, the flashing brine stream, and the condensate stream. The flashing brine stream is carried in two parallel pipes to avoid problems in distributing the brine flow laterally across the module width, as it flows from one module to the next. Expansion joints are provided in all pipes between modules to accommodate thermal expansion and contraction.

Each module is divided into stages by bulkheads, with each stage containing channels for the flashing brine and the condensate and knitted mesh demisters to strip droplets of brine from the vapor enroute to the condenser tubes. Properly sized orifices between stages regulate the flow of flashing brine and product from stage to stage. The orifices are installed in liquid-leg U-tubes at the high-pressure end of the plant to provide sufficient submergence to prevent blowing the liquid seals. The condenser tubes extend in continuous lengths through all the stages in each module, so that only two water boxes and tube sheets are needed for each module.

The heat rejection module has the same basic cross-sectional size and shape as the heat recovery modules, but is only 63 feet long and contains three stages. Construction is basically the same, and the flashing brine and product channels are continuations of their counterparts in the heat recovery modules. The condenser coils, however, are supplied with a cooling water stream separate from the recirculating brine stream in the heat recovery modules.

The brine heater is a separate vessel in which the recirculating brine from the first stage preheater is brought to its final temperature by the externally supplied heating steam. The carbon steel brine heater is 46 inches in diameter, with tubes 15'-6" long.

All heat exchange tubes throughout the plant are of 90-10 copper-nickel construction; tube sheets are of solid copper-nickel or copper-nickel clad over carbon steel; water boxes are copper-nickel for temperatures above 180 F, and carbon steel for lower temperatures. Tubing specifications are detailed in the MSF Plant Equipment List, Section 18.1.

All pumps in the MSF plant are driven with electric motors, and those handling water near its flashing point are vertical-type set in pits to avoid cavitation problems. All pumps, except the brine recycle pumps, are provided with an installed spare to minimize the probability of plant shutdown because of the need for pump maintenance. The two brine recycle pumps are designed to operate in parallel, with each pump capable of providing 55 percent of the required brine recirculation flow at the design head. Thus, if one pump were shut down, the other could continue to operate the plant at a reduced throughput.

The reference MSF plant design provided for a separate deaerator and decarbonator. To make the MSF design comparable to the VTE, Fluor combined the two separate units into a single deaerator-decarbonator vessel 15'-4" in diameter and 26 feet long, with spray inlet nozzles and "Mas Pac" polypropylene packing. The vessel is maintained at 26" Hg vacuum and provided with a flow of stripping steam to remove dissolved oxygen and carbon dioxide from the makeup feed. A two-stage steam jet air ejector system with barometric condensers is provided to remove noncondensable gases from the evaporator vessels and the deaerator-decarbonator. Arrangement of the mechanical equipment, piping, and instrumentation is shown in the Mechanical Flow Diagram, Drawing 4-013. The Chemical Flow Systems and the Electrical Diagrams are shown in Drawings 4-005 and 4-014, respectively.

### 11.0 - TECHNICAL SUMMARY - 2.5 MGD PLANTS

### 11.3 Study Procedure

The source material furnished to Fluor by OSW for plant comparison consisted of a report in proposal form from Stearns-Roger Corporation covering the design and costs of two 2.5 MGD VTE Desalination Plants, one with smooth tubes and one with enhanced tubes; and Volumes I, II-I, III, III, and IV of a Report Manual for a 2.5 MGD MSF Universal Desalting Plant. Both designs covered principally on-site equipment.

### Additions Made by Fluor to All Plants

In order to obtain complete costs, the following off-plot items were added to all three plants:

- 1. A 1000-foot long submarine intake line, an intake-outfall structure, and seawater intake pumps were added as part of the plant equipment necessary for an open coast location.
- 2. Facilities were added to chlorinate all seawater entering the plant, to control biological growth.
- 3. Pumps were added to return the heating steam condensate to its source.
- 4. Buildings, roads, etc. were added as shown on the plot plan and noted on the Equipment Lists.

### Changes Made by Fluor to the VTE Plants

One process change was made to the VTE plants; the sulfuric acid dosage rate to the make-up feed was reduced from 135 ppm to 120 ppm to put consumption on the same basis as the MSF plant.

Several on-site physical changes and additions were made to the VTE plants to place them on a basis comparable to the MSF plant. The principal change was to increase the diameter of the deaerator-decarbonator from 5 feet to 13.5 feet, to provide adequate capacity for removal of inert gases. The size was based on the same criteria used in the design of the deaerator-decarbonator for the MSF plant, and verified by conventional design practice in other plants with which Fluor is familiar.

Other on-site physical changes to the VTE plants included the addition of a caustic mixing tank, and the addition of several pumps to obtain the same reliability in the VTE plant as in the MSF plant. These additions are noted in the Equipment Lists in Section 17.

### 11.3 Study Procedure (Cont'd.)

### Changes Made by Fluor to the MSF Plant

The MSF plant design selected by Fluor to compare with the VTE plants was one of a group of designs supplied to cover a wide range of operating conditions and requirements. This particular selection (Universal Plant Design No. 4) was made because its conditions most nearly corresponded with those of the VTE plant. It was still necessary, however, to make adjustments to bring the MSF plant to the same bases as the VTE plants. This was done in all cases by changing the MSF process design to correspond with or be comparable to the VTE process design. The process changes made to the MSF plant design are:

- 1. The assumed seawater temperature, which had been 85 F, was set at 65 F. This corresponds to an average seawater temperature in Southern California.
- 2. The product water temperature, which had been 95 F, was reduced to 79 F to correspond to the lower temperature of the cooling water.
- 3. The brine concentration ratio, which had been 1.735, was increased to 2.0.
- 4. The fouling factor in the heat recovery stages, which had been .0003, was increased to .0005.
- 5. The fouling factor in the heat rejection stages, which had been .0005, was increased to .0007.
- 6. The heating steam condensing temperature, which had been 265 F, was raised to 267 F.

Using this input, Fluor made a computer run to determine the new design for the MSF plant. This updated computer run is reproduced in the Appendix of this report.

### 11.3 Study Procedure (Cont'd.)

In addition, the following physical changes were made to the MSF plant:

In the Universal Plant Design No. 4, the deaerator and decarbonator are separate vessels. Fluor combined these into a single vessel to correspond with the VTE design, with size and cost of the deaerator-carbonator on each plant in proportion to the relative flow rates through the two vessels. (Enlargement of the VTE deaerator-decarbonator is discussed elsewhere in this section.)

The surface-type ejector condensers specified in the Universal Plant Design No. 4 were changed to barometric type similar to that specified in the VTE plant.

# 11.0 - TECHNICAL SUMMARY - 2.5 MGD PLANTS

### 11.4 - CAPITAL COST SUMMARY

	VTE Plant Smooth Tubes	VTE Plant Enhancèd Tubes	MSF Plant Smooth Tubes
Developed site cost:			
Land (for water plant) Site development and offsite facilities Subtotal, Developed site cost	\$ 160,000 1,028,500 \$1,188,500	\$ 160,000 1,028,500 \$1,188,500	\$ 200,000 1,080,900 \$1,280,900
Desalination plant cost (erected):			
Tube bundles )  Evaporator shells)  Other plant equipment Subtotal, Desalination plant cost	\$1,589,938(1) 1,483,662 \$3,073,600(3)	\$1,412,031(1) 1,440,069 \$2,852,100(3)	\$ 622,300(2) 1,322,500 1,177,700 \$3,122,500
Engineering design, procurement, inspection, etc.	\$ 256,000		\$ 256,000
Interest during construction	\$ 169,400	\$ 161,100	\$ 174,700
GRAND TOTAL ERECTED COST - 1968 DOLLARS	\$4,687,500	\$4,457,700	\$4,834,100

<sup>(1)</sup> Breakdown not available (2) Includes cost of separate brine heaters (3) See Section 15.1 for breakdown

11.5 PRODUCT WATER COST SUMMARY - 2.5 MGD PLANTS

	VTE Plant Smooth Tubes	VTE Plant Enhanced Tubes	MSF Plant Smooth Tubes
Total Capital Cost - 1968	\$4,687,500	\$4,457,700	\$4,834,100
- Escalated to 1970 (1)	\$4,900,000	\$4,680,000	\$5,060,000
Annual Cost, Dollars  Capital charges (2) -includes interest,			
amof bización, inverim repracement and insurance	\$ 307,000	\$ 293,000	\$ 317,000
Utilities (3) Steam Electricity	298,730 43 <b>,</b> 600	298,730 52 <b>,</b> 800	307,460 56,700
Materials and Supplies (3) Chemicals (4) Misc. Maintenance Supplies	37,720 25,000	37,720 25,000	55,760 25,000
Labor, including overhead (3) Operation Maintenance	78,000 48,100	78,000 48,100	78,000 48,100
Total Annual Cost, 1968	\$ 838,150	\$ 875,350	\$ 888,020
Annual Net Water Production, 1000 Gallons (5) Water cost per 1000 gallons - 1968. Water cost per 1000 gallons - Escalațed to 1970 (1)	817,000 \$1.03 (1) \$1.08	817,000 \$1.02 \$1.07	819,000 \$1.08 \$1.14

<sup>2.4%</sup> escalation/yr For fixed charges see Section 13.2 For breakdown see Section 15.0 Includes continuous chlorination of seawater

VTE net production is less because of more in-plant usage.

### 12.0 - DEVELOPMENT POTENTIALS - 2.5 MGD PLANTS

As was the case with the large (250 MGD) desalination plants, there are several areas with the 2.5 MGD plants where potential savings could probably be realized with suitable study and development. These major areas are listed and discussed in the following paragraphs.

### Substitution of Seawater Wells for Submarine Seawater Intake Lines

Although seawater intakes located in deep water on sheltered inlets or tidal estuaries are relatively simple to construct, intake structures along open shores and beaches are generally complicated and expensive. They have to extend for relatively long distances from shore so the intake will be in water deep enough to avoid interfering with shipping and be reasonably free of suspended sand and silt. To protect the pipe from storms and erosion, it must be placed in a trench in the ocean floor and covered - an expensive procedure. In fact, the submarine seawater intake systems for the 2.5 MGD plants sited in Southern California have proven to be one of the more costly portions of the whole plant structure. The submarine type of intake was chosen for this small-plant study in spite of its high cost because it is a proven method for this area. Many of the power generation stations along the Southern California coast obtain their condenser cooling water in this manner, and have found that the submarine line does provide a reliable source of fairly clean seawater. One disadvantage is the tendency of the submarine line to collect marine growth, such as mussels, barnacles, and seaweed. In the power plant conduits, marine growth is minimized or eliminated by chlorination or by intermittent heat-shocking of the entire intake line.

Seawater wells are being successfully used in some areas as an alternative to the submarine intake line. Where the underground strata is a permeable material such as coral, drilled wells can provide an adequate supply of filtered seawater for power plant cooling and for desalting plant feed water. The water from such wells is constant in temperature and is completely free from sand, silt, and marine life. Moreover, the cost of drilling seawater wells is many times less than that of building a submarine intake line. It has been suggested that even where the subsurface stratum is not very permeable, several vertical gravel-packed wells, or a Ranney well with gravel-packed horizontal bores radiating from a central vertical bore, could produce sufficient clean seawater at a cost less than that of the submarine line. A seawater collector of the latter type is successfully operating at Ventura, California.

For small desalination plants located on open shores, there is a good possibility that seawater wells could provide an inexpensive method of obtaining cool, filtered seawater devoid of marine life and needing no chlorination or screening. Because submarine intake systems constitute a significant fraction of the capital cost of the 2.5 MGD plants considered in this study, seawater wells could make a significant reduction in the capital cost of such plants.

### Increased Use of Plastics

The reference VTE plant designs specify plastic (fiberglass reinforced epoxy or polyester) pipe for most of the seawater and brine piping, whereas the reference MSF plant design specified cement-lined carbon steel or extra strong carbon steel or 90-10 Cu-Ni pipe for these streams. In addition, the VTE reference design uses plastic for 9 of the 14 water box covers over the falling-film elements.

### 12.0 - DEVELOPMENT POTENTIALS - 2.5 MGD PLANTS (Cont d.)

At the present time, plastic pipe is obtainable up to 30 inches in diameter, but experience is limited in the use of plastic pipe larger than about 18 inches. Moreover, in all sizes the plastic materials must be carefully selected to conform to their pressure and temperature limitations. Because there is still so little experience in the installation of plastic pipe, installers must be thoroughly trained and supervised in order to avoid costly rework.

Fluor believes, however, that as experience is gained in the manufacture and installation of plastic pipe, significant first cost and maintenance savings are possible through the greater use of plastics, especially in the case of MSF plants.

### Longer Tubes for MSF Plants

At present, heat exchange tubes up to about 100 feet long are available. They can be shipped or barged in straight lengths to the site for field installation, or they can be shop-installed into evaporators, which are then shipped to the site.

The use of even longer tube lengths would eliminate a few tube sheets and water boxes, plus the piping which now interconnects a greater number of shorter evaporators. For very large water plants it is conceivable that tubes of any length can economically be either fabricated in the field or shipped in rolls and then straightened in the field. For small plants (in the 2.5 to 5 MGD range), this means of obtaining longer tubes would not be expected to result in a cost saving. However, it is probable that there are, currently, several tube mills and many potential plant sites so located that straight tube lengths or shop-tubed evaporators up to 150 feet long could be shipped. This would result in a small but significant saving for an MSF plant.

### Use of Enhanced Tubes for MSF Plant

There is currently a great deal of interest and activity in the development and testing of enhanced surface tubes designed to improve heat transfer between liquid and vapor phases, such as is found in the tubes in MSF desalination plants. In the portion of this study dealing with the large (250 MGD) desalination plants, it was shown that the use of enhanced-surface tubes in the MSF plant would be potentially advantageous over the use of smooth tubes (Sec. 4.0). Presumably this same advantage would apply to a lesser extent in the small MSF plants; however, this aspect was not investigated in the current project.

In the study of the large MSF plant, the prices and performance of the spirally-corrugated tube of a single manufacturer (Wolverine Tube) were evaluated. It is known that somewhat different configurations are under study by Oak Ridge National Laboratories, by Olin Brass, and perhaps by others. It is anticipated that the next few years may see the development of improved performance and reduced manufacturing costs of enhanced surface tubes for such applications, which should be reflected in reduced capital costs of MSF plants.

### 12.0 - DEVELOPMENT POTENTIALS - 2.5 MGD PLANTS (Cont'd.)

### Use of Turbine Drives for Large Pumps

The ground rules for this study stipulated the use of electric motor drives for all major pumps. This implies a dual-purpose plant, since electric motor drives on large pumps usually show an advantage only where high-pressure steam is already being used to generate electricity on-site. For single-purpose plants, it is generally more economical to drive the large pumps with high pressure steam turbines, using the turbine exhaust for feed water heating. It is expected that steam turbine drives on major pumps could reduce operating costs on a single-purpose plant, and their use would lead to lower water costs, particularly with the MSF plant.

### Increased Maximum Brine Temperatures

For any single-purpose water evaporation plant, the use of higher maximum brine temperatures offers a potential for increasing the performance ratio and thus lowering product water cost. A recent study by Bechtel Corporation for OSW (R&D Progress Report No. 175) found 350 F to be the optimum brine heater temperature for large MSF plants. At this brine temperature, the use of additional flash stages would allow the plant to produce more product with a given thermal input. Similar savings could be expected in VTE plants by increasing the number of stages and effects.

In order for a plant to operate at such elevated temperatures on seawater feed, a practical and inexpensive way must be developed to eliminate the formation of scale on heat transfer surfaces. Even with acid treatment, temperatures much above 260 F cannot be achieved without the risk of scaling. Calcium sulfate scale, which appears above 250 F, is not inhibited by acid treatment of the feed, nor can it usually be removed by anything except mechanical means.

A promising method of scale elimination has recently been developed by the W. R. Grace Company, which removes a major portion of the calcium ion in the seawater feed by precipitating it as calcium carbonate. It is called the lime-magnesium carbonate (LMC) process. Since the calcium is precipitated as calcium carbonate, the process also removes bicarbonate and carbonate ions, and eliminates the necessity for acid treatment of the feed.

This LMC pretreatment process is currently under test at the OSW West Coast Test Facility in Chula Vista, where it is treating the feed to the Clair Engle desalination plant. Early reports indicate that the plant is performing well. If the LMC water pretreatment method should prove to be effective and economical, two-fold savings would result. The significant cost of acid addition would be eliminated, and higher brine temperatures would allow overall plant efficiency to be improved.

### 13.0 - BASES FOR COST ESTIMATES - 2.5 MGD PLANTS

### 13.1 Bases for Capital Cost Estimate - 2.5 MGD Plants

### General Comments

The basic purpose of the estimates of the VTE and MSF 2.5 MGD plants is to compare the capital costs and the water costs for these two types of plants. Both estimates are based on A&E type designs, in sufficient detail for obtaining lump sum proposals. In the following paragraphs specific portions of the capital cost estimate are discussed, with comments, where appropriate, on the assumptions and method of approach to each.

### Scope

The capital cost estimate for the 2.5 MGD desalination plant assumes a single purpose plant and includes only the water plant. The heating steam source is assumed to be a boiler plant adjacent to the water plant. The cost of the boiler plant is not included in the capital costs of the water plants. Its cost would be the same for either type of water plant.

The following procedure was used for estimating the capital costs of the 2.5 MGD plants. The quoted plant selling price given in the source report was accepted as being a correct estimate of the cost of the VTE plant as defined in the report. To this selling price Fluor added: (a) 5% price escalation; (b) the estimated cost of facilities shown on the plot plans which are outside the quoted plant; (c) the cost of added or changed equipment within the quoted plant; and (d) owner's costs.

For the MSF plant the capital costs were estimated completely by Fluor. Equipment prices were based on quotes or in-house information. Foundations, piping, electrical, instrumentation, etc. were estimated and material take-offs made from the flow sheets and drawings provided in the reference design.

### Source for Process and Mechanical Designs

For the 2.5 MGD conventional (smooth-tube) and enhanced-tube VTE plants, the basis for design is a report in proposal form submitted to Union Carbide Corporation by Stearns-Roger Corporation. This report is entitled "2,500,000 GPD Multiple Effect Falling Film Desalination Plant," Project No. B-35298, and is dated September 30, 1968, with an addendum dated November 1968. For the 2.5 MGD MSF plant the design basis is the "Report on Design of a 2.5 Million Gallon per Day Universal Desalting Plant," prepared for the Department of the Interior, Office of Saline Water, by Burns and Roe, Inc. This report is dated June 1967.

### 13.1 Bases for Capital Cost Estimate - 2.5 MGD Plants (Cont'd.)

### Site Preparation

The plants are assumed to be located on a Southern California coastal site, with a 200-ft. wide beach rising to an elevation of 35 ft. at the base of the coastal cliff. At the cliff the terrain is assumed to rise 50 ft. to a level plateau at 85 ft. elevation, which extends inland beyond the site limits. The prepared site is to be at elevation 35 ft. An allowable soil bearing pressure of 3500 psf is assumed.

A major expense in site preparation is excavation and grading. For the VTE plant, 71,000 cubic yards must be removed; for the MSF plant, because of its greater area, the figure is 84,000 cubic yards. It is assumed that this excavated material could be used as fill on a location adjacent to the plant site.

### Buildings

The cost of a control building, switchgear building and a small shop is included in the cost of each plant.

### Owner's Costs

The following owner's costs are included in the capital costs of both the VTE and MSF plants.

- a. Cost of land occupied by Water Plant assumed at \$100,000 per acre.
- b. A&E costs, permits, etc.
- c. Owner organizational costs.
- d. Interest during construction (includes bond financing costs).
- e. Startup costs labor, chemicals, etc.

### Product Water Treatment

For both the VTE and the MSF plants the product water is assumed to be delivered at the plant boundary under a pressure of 10 psi. No capital costs are included in this estimate for product water treatment or storage.

### 13.1 Bases for Capital Cost Estimate - 2.5 MGD Plants (Cont'd.)

### Price Level

For the MSF plant, labor and equipment pricing is based on 4th quarter 1968 prices, then escalated to 1970 level based on an 18-month engineering-construction schedule. The VTE plant prices are assumed to be also based on 4th quarter 1968 prices. The reference quotation is subject to 5% escalation to bring it to the same time frame as the MSF plant, and this has been added. The facilities added by Fluor are escalated by Fluor on the same basis as for the MSF plant. Annual escalation from 1968 to 1970 is assumed to be at the same rate as the average annual escalation from 1958 to 1968 in Southern California. Escalation is estimated to have averaged 4.2% annually for labor and 2.1% for material. It should be noted that escalation rates during the last five years are much higher than the ten-year average.

### Labor Availability

Except for some shift work and overtime by concrete workers and riggers, a 40-hour work week is assumed. Travel and subsistence allowances to the site are included.

### Equipment Pricing

Only U. S. manufacturers were contacted for equipment priced by Fluor.

### Spare Parts and Shop Tools

The cost of spare parts and shop tools is not included in the capital costs.

### Taxes

It is assumed the owner will be tax exempt and, therefore, the California State sales tax of 5% on materials is not included.

### Form of Estimate

The breakdown is presented in essentially the form and detail shown in Appendix F of the 1965 Saline Water Conversion Report.

### 13.0 - BASES FOR COST ESTIMATES - 2.5 MGD PLANTS

### 13.2 Bases for Product Water Cost - 2.5 MGD Plants

### Escalation

It is assumed that the plant could be operational in mid-1970 if started in early 1969. Product water costs are estimated on the basis of the purchasing power of 1968 dollars, and operational costs are then escalated to 1970, using the average annual escalation experienced from 1958 to 1968. Escalation is estimated at 4.2% annually for labor and 2.1% for material. It should be noted that escalation rates of the last five years have been much higher than for the ten-year period.

### Cost of Money

Municipal bond financing is assumed. Bond interest rate is taken as 4.25%.

### Cost of Bond Financing and Interest During Construction

Underwriters and bond counsel fees and interest during construction are included in the capital costs.

### Repayment Period

Investment costs are amortized over a 30-year period corresponding to the estimated life of the desalination plants. The annual amortization factor of 1.71% is based on a sinking fund repayment method over a 30-year period at 4.25% interest.

### Interim Replacement

Both the VTE and the MSF plants are expected to have a useful life of 30 years; however, a 0.35% factor is included for replacement of portions of the plants not expected to last the full 30 years. The 90-10 Cu-Ni tubing of each of the plants is assumed to have a 30-year life.

### Fixed Charges on New Plant Costs

The total annual fixed charge is based on the following percentages of capital investment:

a.	Cost of money	4.25%
b.	Bond amortization factor (30-year basis)	1.71%
С.	Plant replacement factor	None
d.	Plant interim replacement factor	0.35%
e.	Tube interim replacement factor	None
f.	Property damage insurance	.25%
g.	Taxes	<u>None</u> 6.56%

### 13.2 Bases for Product Water Cost - 2.5 MGD Plants (Cont'd.)

### Energy Costs

Energy Costs used in estimating water costs were specified by OSW and are:

	VTE Plant	MSF Plant
Steam supplied at plant boundary:		
150 psig <b>-</b> ¢/10 <sup>6</sup> Btu	50	50
42 psia – ¢/10 <sup>6</sup> Btu	50	-
40 psia – ¢/10 <sup>6</sup> Btu	-	50
Electric power - 4160 V. at plant boundary - ¢/Kwhr	0.85	0.85

### <u>Chemical Costs</u> (delivered to site)

Price,	100% Basis	VTE Plant	MSF Plant
Sulfuric acid (93%)	\$/ton	30.00	30.00
Caustic soda - 50% solution	\$/ton	150.00	150.00
Chlorine - 100%	\$/ton	130.00	130.00
Anti-foam solution - 100%	\$/lb	.40	.40

### Maintenance Materials and Supplies

The annual cost of plant maintenance materials and supplies is estimated at 0.6% of direct capital cost.

### Insurance

The annual insurance premium on the desalination plant is estimated to be 0.25% of capital cost.

### Taxes

It is assumed that the owner will be a tax-exempt municipal or governmental agency, and that no city, country, state or federal tax or tax equivalent payments are applicable.

### Load Factor

It is assumed the plants will be operating at full design load (2.5 MGD) all of the time that they are on stream.

### 13.2 Bases for Product Water Cost - 2.5 MGD Plants (Cont'd.)

### On-Stream Factor

For a single purpose plant, it is assumed that the plant will be available or on-stream 90% of the time. It is assumed to be shut down for maintenance, repairs, power plant outages, etc., the remaining 10% of the time.

### Average Production Rate

Since the maximum daily gross capacity of each plant is 2.5 MGD, the yearly average capacity is 2.25 MGD (90% of 2.5). The net capacity is the gross capacity less plant useage for bearing cooling, glands, flushing, etc.

### Product Water Pumping Requirements

Product water is delivered to the plant boundary at 10 psig and 90 F. No other pumping, passivation, or handling charges are included, since conveyance pipeline and storage facilities are not considered plant costs.

### Operation and Maintenance Labor

Operation and maintenance labor costs are based on a single staff and a single control room for both steam and water plants. Personnel and estimated cost are shown in Section 15.3.

### Owner's Administration and General Costs

These costs are estimated at 30% of operation and maintenance labor costs. The 30% includes payroll burdens of 20% (vacations, sick leave, State Disability Insurance, Federal Insurance Compensation Act, and Unemployment Compensation Disability Benefits).

### Interest on Working Capital

Interest on working capital is included in the Administration and General portion of operating costs.

### 14.0 - SITE - 2.5 MGD PLANTS

For the purposes of this comparative study, all of the small (2.5 MGD) plants are assumed to be sited at the same hypothetical Southern California coastal location as was assumed for the large plants. The beach area is 200 feet wide, rising gradually from the breaker line to an elevation of 35 feet as the base of the coastal cliff. At the cliff the terrain rises abruptly to 85 feet elevation, and is assumed to then extend inland for an indefinite distance at 85 feet above MLLW.

The site will be excavated to an elevation of 35 feet for placement of the evaporation plant, access roads, and service and storage facilities. Typical soil in this region is readily excavated, yet is competent to provide good foundation support. An allowable soil bearing pressure of 3500 psf is used, and cut banks are assumed to be stable at an angle of 45°.

Land transportation facilities are conveniently available to the site. U. S. Highway 101 and the Santa Fe railroad both parallel the coastline in this general area, and overland access to the site presents no problems. No dock or harbor facilities are presently available in this area to accommodate water transport. However, water shipment of large items could be accomplished with the use of beaching-type craft and provision of the necessary unloading equipment.

The intake structure containing the seawater pumps and traveling screens is located at approximately the original breaker line, and the nearest portion of the plant proper about 225 feet inland from this. A buried intake pipe extends seaward 1000 feet to the seawater intake caisson and fish cap. This will be a 39-inch reinforced concrete line for the VTE plants, and a 42-inch line for the MSF plant. The top of the fish cap will be at least 15 feet below MLIW to avoid navigation hazards, and well above the ocean floor to avoid entraining silt with the feed stream. Reject water from the evaporation plant will be piped to a point adjacent the seawater pumps and released into the surf through an open channel.

This study is limited to consideration of the facilities required for the water plant, and does not include any site allowance for boiler plant, water storage, or any other related facilities. The water plant site plans are shown on Drawing 002 for the smooth-tube VTE plant, Drawing 001 for the enhanced surface tube VTE plant, and Drawing 011 for the MSF plant. Each plant will require a plot about 125 feet deep. The VTE plants are approximately 215 feet wide while the MSF plant needs 275 feet.

### 15.1 Capital Cost Breakdown

		_	TE Plant ooth Tubes	_	TE Plant nced Tubes	_	ISF Plant oth Tubes
I.	land	\$	160,000	\$	. 160,000	\$	200,000
II.	Site Development and Offsite Facilities						
	<ol> <li>Mass excavation</li> <li>Roads, surfacing, etc.</li> <li>Buildings</li> <li>Submarines pipelines</li> <li>Seawater intake and outfall structures</li> <li>Screens, seawater pumps for intake structure</li> <li>Chlorination system (incls. pumps)</li> <li>Utilities (service water, service air, sewers</li> <li>Subtotal - Section II</li> </ol>		45,700 8,000 28,700 550,000 124,100 196,500 49,100 26,400	<del></del> -	45,700 8,000 28,700 550,000 124,100 196,500 49,100 26,400	<del></del>	53,300 8,400 37,400 560,700 124,100 221,500 49,100 26,400 ,080,900
		*-	.,, )	*	, ,,	-	,
III.	Conversion Plant						
	<ol> <li>Evaporators         <ul> <li>a. Bundles</li> <li>b. Shells</li> </ul> </li> <li>Brine heater</li> <li>Noncondensable removal equipment</li> <li>Pumps and motor drivers</li> </ol>					1	601,300 ,322,500 21,000 20,800
	<ul><li>a. Blowdown</li><li>b. Brine recycle</li><li>c. Product water</li><li>d. Condensate</li></ul>						14,800 91,000 13,600 6,300
	<ul> <li>5. Make-up Pretreatment Systems</li> <li>a. Chemical feed systems (incl. pumps, tanks and piping)</li> <li>b. Deaerator-decarbonator</li> <li>6. Excavation, foundations and miscellaneous</li> </ul>	3					52,100 69,300
	steel						255,200
	<ul> <li>7. Piping</li> <li>a. Seawater and brine</li> <li>b. Product water</li> <li>c. Steam and condensate</li> <li>d. \Noncondensables</li> <li>8. Electrical</li> </ul>						223,000 18,100 8,700 41,100 145,300
	<ul> <li>Instrumentation</li> <li>Instruments, controls and control panel</li> <li>Instrument air system (compress., receiv</li> </ul>						121,200
	driers, pipings, etc.)  10. Insulation  11. Painting  12. Spare Parts	· ,	<u>None</u>		None	_	18,100 61,300 17,800 None
	* VTE conversion plants estimated by others ** Additions to VTE plants estimated by Fluor		2,698,100 375,500	*2	2,476,600 375,500		
	Subtotal - Section III	\$3	3,073,600	\$2	2,852,100	\$3	3,122,500
	Management and Inspection, Owner Organization Costs, and Owner Startup Costs		256,000		256 . 000		256,000
v.	Interest During Construction	\$	169,400	<u>\$</u> _	161,100	<u>\$</u> _	174,700
	GRAND TOTAL - 1968 DOLLARS	<b>\$</b> 2	,687,500	\$1	1,457,700	\$4	,834,100

<sup>\*</sup> These are Stearns-Roger estimated selling prices plus their stated maximum escalation of 5%. See "A Report in Proposal Form" prepared by Stearns-Roger for Union Carbide dated September 30, 1908.

<sup>\*\*</sup> For breakdown, see Section 15.4

### 15.2 Product Water Cost Breakdown

### 1968 Production Costs

Daily Production On Stream Factor Gross Production Prod. Water Usage in Net Production	MGD % MGY Plant MGY MGY	Smooth Tubes  2.5 90 822 5 817	Plant Enhanced Tubes  2.5 90 822 5 817	MSF Plant Smooth Tubes  2.5 90 822 3 819
Steam - Saturated Heating Steam	lbs/hr x 10 <sup>3</sup> Btu/hr x 10 <sup>6</sup> ¢/Btu x 10 <sup>6</sup> \$/hr \$/day \$/yr	80.2 74.7 50 37.3 896 \$294,000	80.2 74.7 50 37.3 896 \$294,000	80.9 75.6 50 37.8 910 \$298,000
Ejector Steam Subtotal - Steam	lbs/hr x 10 <sup>3</sup> Btu/hr x 10 <sup>6</sup> ¢/Btu x 10 <sup>6</sup> \$/hr \$/day \$/yr \$/yr ¢/1000 gals	1.0 1.2 50 .60 14.40 \$ 4730 \$298,730 36.6	1.0 1.2 50 .60 14.40 \$ 4730 \$298,730 36.6	2.0 2.4 50 1.20 28.80 \$ 9460 \$307,460
<u>Power</u> Subtotal - Power	kw ¢/kwh \$/hr \$/day \$/yr ¢/1000 gals	650 0.85 5.52 1 <u>73</u> \$ 43,600 5.3	785 0.85 6.70 <u>161</u> \$ 52,800 6.5	845 0.85 7.20 <u>173</u> \$ 56,700 7.0
Subtotal - Energy	<b>\$</b> /yr ¢/1000 gals	<b>\$</b> 353 <b>,33</b> 0 41 <b>.</b> 9	<b>\$</b> 362 <b>,</b> 530 43.1	<b>\$</b> 364 <b>,</b> 160 44.5

### 15.2 <u>Product Water Cost Breakdown</u> (Cont'd.)

### 1968 Production Costs (Cont'd.)

	VTE	E Plant	MSF Plant
	Smooth Tubes	Enhanced Tubes	Smooth Tubes
Chemicals			
Acid - 66° Be (93%) tons/day	2.2	2.2	2.7
\$/ton	30	30	30
\$/day	66	66	81
\$/yr	\$ 21,700	<b>\$</b> 21,700	\$ 26,600
Caustic - 50% Solution tons/day	.10	.10	.12
\$/ton	50	50	50
\$/day	5	5	6
\$/yr	\$ 1,640	\$ 1,640	<b>\$</b> 1,960
Chlorine - 100% tons/day	.26	.26	.37
\$/ton	130	130	130
\$/day	34	34	48
\$/yr	\$ 11,100	\$ 11,100	\$ 15,700
Miscellaneous Chemicals Anti-foam \$/day Ferrous sulfate \$/day \$/yr Subtotal - Chemicals \$/yr \$/1000 gals	None	None	25
	10	10	10
	\$ 3,280	<b>\$</b> 3,280	<b>\$</b> 11,500
	\$ 37,720	<b>\$</b> 37,720	<b>\$</b> 55,760
	4.6	4.6	6.8
Miscellaneous Maintenance Materials  Charts, rags, gaskets, lube oil, paint, fittings & other supplies  Subtotal - Misc. Material \$/yr	<b>\$</b> 25 <b>,</b> 000	<b>\$</b> 25 <b>,</b> 000	<b>\$</b> 25 <b>,</b> 000
$\phi/1000$ gals		3.1	3.1
Operating & Maintenance Labor			
Operation \$/yr Maintenance \$/yr Supervision \$/yr Administration & General 30% Subtotal - O&M \$/yr \$/1000 gals Grand Total - Water Production Costs \$/yr \$/1000 gals	31,000 16,000 29,100 \$126,100 15.5	50,000 31,000 16,000 29,100 \$126,100 15.5 \$551,350 66.3	50,000 31,000 16,000 29,100 \$126,100 15.5 \$571,020 69.9

<sup>1</sup> Excluding fixed charges on capital cost

### 15.3 Operating and Maintenance Cost Estimate for a 2.5 MGD Single Purpose VTE or MSF Plant<sup>1</sup>

	Duty <u>Requirement</u>	Personnel Required	1968 Annual Rate So. Calif.	Annual Cost		
Plant Superintendent	5 days - 8 hrs.	1	<b>\$</b> 16 <b>,</b> 000	\$16,000		
Operators	7 days - 24 hrs. (1 per shift)	4 <del>1</del> 2	10,000	45,000		
Plant Chemist	5 days - 4 hrs.	<u>5</u>	10,000	5,000		
Maintenance Technicians	5 days - 16 hrs.	2	12,000	24,000		
Janitor	5 days - 8 hrs.	$\frac{1}{10}$	7,000	7,000 <b>\$</b> 97,000		
Administration and General, including payroll burdens and fringe benefits - 30%						

<sup>1</sup> Personnel would also operate and maintain the steam generation plant, but the entire cost is charged to the water plant, since it is a single-purpose plant.

### 15.4 Cost Breakdown of Additions to VTE Plants 1

1.	Pumps	
	a. Nondeaerated seawater feed P-lA & B	\$ 40,500
	b. H. R. distillate pump spare P-41B	1,000
	c. Condensate return P-53A & B	8,100
	d. Spare for P-24 and 25 P-245	5,200
-2.	Larger deaerator	28,200
3.	Caustic mixing tank with agitator	4,200
4.	Larger instrument air compressors and system	10,700
5.	Excavation, backfill and foundations for above	31,200
6.	Additional piping and supports	121,900
7.	Additional electrical	90,700
8.	Additional instrumentation	28,100
9.	Additional insulation	1,200
10.	Additional painting	4,500
		\$375,500

These additions, made to both VTE plants, were considered necessary by Fluor to insure proper plant operability and reliability.

### 16.0 - 2.5 MGD DESIGN - VTE AND MSF PLANTS

### 16.1 Seawater Intake and Outlet

The hypothetical Southern California site chosen for location of the 2.5 MGD plants is typically along an unprotected shoreline, with shallow water extending some distance from the shore. The type of water inlet structure commonly used by power plants in this area is a submerged pipeline extending a considerable distance into the ocean. The water entrance must be in deep enough water so that it can be well above the ocean floor to prevent drawing in sand and silt during storms, and still far enough below the water surface to present no hazard to navigation. It is assumed in this case that this will require about 1000 feet of inlet line beyond the surf.

The intake structure will be located near the surf line, and will include traveling screens to remove debris from the water, screen cleaning equipment, and the seawater intake pumps. The discharge line, returning concentrated, warm brine to the sea, is an open channel releasing the brine into the surf. Associated with the intake line is a small conduit carrying heavily chlorinated seawater to be released in the mouth of the inlet pipe.

This inlet structure and submerged pipeline is an expensive construction project, and there may be other methods of obtaining clean seawater that would be less costly. However, this is a tried and proven method, and for this comparative study it was assumed that all these 2.5 MGD plants would use this type of inlet.

### 16.0 - 2.5 MGD DESIGN - VTE AND MSF PLANTS

### 16.2 Chemical Systems

In common with the 250 MGD plants, the small VTE and MSF plants require the addition of chemicals to the system to inhibit biological growth in the plant conduits, prevent the deposition of scale, prevent corrosion, and eliminate foaming. Chemical flows for both plants are in Drawing 005.

Chlorine addition is required to prevent the growth of marine life within the plant structure. The chlorine must be added at the extreme outer end of the inlet pipe, to keep this pipe from becoming fouled with marine growth.

Chlorine is purchased as a liquefied, compressed gas, and must be dissolved in the water in which it will be used. If usage is large, a heat exchanger must be provided to vaporize the liquid chlorine. For plants of this size, however, it is feasible to utilize ambient heat to vaporize the chlorine as it is required. A one-ton cylinder will provide several hundred pounds of gas per day without any special heating means. By manifolding 5 one-ton cylinders together, sufficient gaseous chlorine can be produced to continuously treat the seawater inlet stream with 10 ppm of chlorine, if that rate should be required.

Methods of metering and dissolving the chlorine into the seawater require certain procedures and precautions. Chlorine is soluble in water to large concentrations, but requires time and large contact surface to dissolve. Dry chlorine gas is not corrosive, nor is dissolved chlorine in low concentrations; however, wet chlorine gas and water with high concentrations of dissolved chlorine are quite agressive to most metals. Chlorine gas from the tanks is passed through a pressure reducer and then aspirated into a mixing jet, where it is blended with water to form a solution of approximate 0.1% chlorine. This is then pumped to the end of the submerged intake pipe and released at the inlet of the conduit. In this manner marine growth can be controlled throughout the cold end of the plant. It is the experience of some operators that only intermittent chlorination is required - perhaps a half-hour to an hour once a day. Others find continuous chlorination to be more effective. equipment has been sized to be capable of continuous chlorination. Steel pipe is used to carry the dry chlorine gas, and fiberglass reinforced plastic pipe for carrying the chlorine solution out to the end of the intake line. Hastelloy B is the best material for the difficult environment of the ejector.

Sulfuric acid is used to treat the make-up feed going to the deaerator-decarbonators. It breaks down the bicarbonate content of the seawater into carbon dioxide and water, preventing the deposits of calcium carbonate and magnesium hydroxide on heat transfer surfaces. The acid is injected with a metering pump into the warmed make-up feed leading to the deaerator-decarbonator, the amount being controlled by measuring the pH of the stream. If the pH of the make-up stream is too low after leaving the deaerator, it may be necessary to add caustic to raise the pH to prevent metal corrosion in the system. A caustic addition system is provided, which is quite similar to the acid system, with the caustic injected by a proportioning pump.

### 16.2 Chemical Systems (Cont'd.)

Under some circumstances it has been found necessary in MSF plants to add a small amount of antifoaming agent to the recirculating feed to prevent excessive build-up of froth and foam. A proportioning pump and storage facility has been provided in the MSF plant only, to add antifoamer if it is necessary. It has not been found necessary to use antifoamer in VTE plants.

### 16.0 - 2.5 MGD DESIGN - VTE AND MSF PLANT

### 16.3 Electrical

The initial electrical design of the reference VTE plants called for 480-volt power to be provided at the plot boundary as the sole source of electrical power. Since the VTE plants contain several motors in the 200 to 500 hp category, Fluor changed the plant supply voltage to 4160 volts - a more practical and economical voltage for motors of this size. This also corresponds to the electrical supply provided in the reference Universal MSF Plant. Further details of the electrical distribution system may be found in the Electrical One-Line Diagrams - Drawing 4-006 for the VTE plant and Drawing 4-014 for the MSF.

### 16.0 - 2.5 MGD DESIGN - VTE AND MSF PLANTS

### 16.4 <u>Instrumentation</u>

### General

Except where noted, the instrumentation recommended in this report for both VTE and MSF plants is essentially the same as that proposed by the reference designs. Changes have been made where, in Fluor's opinion, they are needed to improve plant operability, to reduce cost, or to place the instrumentation of the two plants on a comparable basis. Changes made are discussed and reasons are given.

A review of the instrumentation of the 2.5 MGD VTE and MSF plants shows them both capable of adequate process control; however, there is a basic difference in concept between the two designs. In the MSF plant, the designer has placed major indicating and control instruments in the central control room. All normal plant operation and surveillance will be accomplished from the control room, with only certain initial startup and final shutdown operations requiring local manual control. In contrast, the VTE plant designer has the VTE control room serving principally as a monitoring station. Excepting a few major control loops, the VTE controls and corresponding indicating points are in the field. Suitable control can be achieved with either method, but more operators will be required when controls are installed on local panel boards. To place both plants on an equal basis, Fluor has modified the VTE design by moving all field-mounted controllers to the control room. Most thermometers have been replaced by thermocouples connected to a multipoint temperature indicator on the control room panel board.

In contrast to the VTE plant, the MSF plant designer has proposed a very high level of instrumentation. Instruments which Fluor feels are unnecessary have been eliminated in the interests of economy. These include some of the temperature indicators, gage glasses, and conductivity analyzers.

### Materials of Construction

Material of construction have not been specified in the reference designs for the wetted parts of most instruments. Fluor recommends the following materials for instruments exposed to the various plant fluids and the instrumentation was priced on this basis:

- a. For wetted parts exposed to brine and seawater: monel
- b. For product water and noncondensable gases: 316 stainless steel
- c. For steam, condensate, and miscellaneous services: carbon steel

### Instrumentation Diagrams

Instrumentation for each plant is shown on the Mechanical Flow Diagrams - Drawing 044 for the VTE and 013 for the MSF plant.

### 16.4 <u>Instrumentation</u> (Cont'd.)

### Temperature Instruments

Standard pneumatic transmitters are specified for control loops. Thermocouple temperature points are recorded on a potentiometer-type strip chart multipoint recorder, or indicated on a multipoint indicator, using selector switches for point selection.

### Thermocouples and Thermometers

On the VTE plant, Fluor has added thermocouples at the brine pumps, preheaters, and brine stages to provide needed temperature indication from these points in the control room. On the MSF plant, thermometers had been provided on each stage vent crossover line. Fluor has eliminated these, leaving only one thermometer per module on the vent lines.

### Pressure Instruments

Standard force-balance pressure transmitters are specified, using standard materials except where process conditions require stainless steel or monel for corrosion protection. Fluor added a board-mounted pressure indicator with field transmitter to the seawater inlet on the VTE design. A board-mounted pressure indicator and field transmitter were added to the deaerator on the MSF plant.

### Differential Pressure and Flow Transmitters

Standard force-balance type D/p cells are specified, using suitable materials for corrosion protection as described above. On the MSF plant, the differential pressure control across the brine heater outlet control valve has been changed from board-mounted to field-mounted override control. Fluor believes that board-mounted control is not necessary for this service.

### Orifice Plates and Flow Tubes

On the MSF plant, Fluor has changed flow nozzles to monel orifice plates for all brine water service except for the seawater supply and brine recycle, where plastic flow nozzles of the insert type are specified. Stainless steel orifice plates instead of flow nozzles are used on all other fluids of the MSF plant. In the VTE design, Fluor has added an orifice plate and transmitter in the line feeding 150 lb. steam to the ejectors. This makes the two designs comparable in this respect.

### Liquid Levels

For brine and seawater service, Fluor recommends the use of flange-mounted force-balance type D/p cells with stainless steel flanges and monel diaphragms. Fluor believes this type of cell is superior to the external float type instrument for corrosive service. The mounting is

### 16.4 Instrumentation (Cont'd.)

nearly flush, no stagnant areas are created, and the number of wetted parts is minimized. Since fewer corrosion-resistant parts are necessary, significant cost savings can be realized. For less corrosive service, such as product water and condensate, external float type cells are provided.

For comparability of instrumentation, Fluor has moved to the control room all field mounted level controllers in the reference VTE plant. In the MSF plant the level control override for the first stage was changed from control room panel mounted to field mounted.

### Control Valves

On brine service, control valves are to be wafer type butterfly valves if four inches in diameter or larger, and globe valves if smaller than four inches. Materials of construction will vary with the temperature of the fluid. Butterfly valves above 180 F. will have Ni-resist body and disk, and 316 SS shaft, with metal seats (no liner); below 180 F., cast iron bodies, Ni-resist discs, 316 SS shafts, and Buna N rubber lining. Globe valves in brine service are bronze with stainless steel trim.

For condensate, product water, noncondensible gases and stripping steam, control valves are to be wafer type butterfly valves for 4-inch size and larger, with cast iron bodies, Ni-resist discs, 316 SS shafts, and Buna N rubber lining. For sizes smaller than 4-inch, globe valves will be used. Valves 2-inch and smaller will be bronze with stainless steel trim, while  $2\frac{1}{2}$  and 3-inch valves will have iron bodies with bronze trim.

For turbine bleed steam service, all control valves will be globe type. Two-inch and smaller valves will be 600-lb. class forged steel with plug-type stellite discs and seats; larger valves are 150-lb. class carbon steel with stainless steel trim.

### Gage Glasses

Standard materials are specified on all gage glasses except those in brine and seawater services, for which copper-nickel alloy with monel trim valves is specified. On the VTE design, Fluor added gage glasses to all of the evaporator stages. This will allow the water levels to be monitored at the effect pump suctions. On the MSF design, gage glasses on each product water stage were eliminated, leaving only two product water gage glasses per module.

### Relief Valves

Relief valves are standard material and specified as shown on the P&I diagram. They are provided on the discharge side of the positive displacement additive pumps, and on the inlet steam line of each plant. The relief valve on the MSF brine stream at the brine heater was deleted.

### 16.4 Instrumentation (Cont'd.)

### Switches, Alarms, and Solenoid Valves

Dual function pneumatic switches with adjustable contacts are specified for alarm and shutdown. Switches are mounted behind the control panel.

Annunciator alarm panels are solid state with backlighted nameplates and with Test and Acknowledge buttons. Switch contacts open to sound alarm. Each panel is provided with 20% spare annunciator points.

### Panel Instruments

Recorders and recorder-controllers are miniature pneumatic type. Adjustments and servicing can be performed from the front of the panel. Indicators are vertical scale ribbon type.

### Control Panel

The control panel is a standard vertical standing type with semigraphic diagram of the plant process.

### Analyzers

Conductivity analyzers are provided only on the final product water streams. Other analyzers on the MSF plant were deleted as being not essential for plant operation. Analyzers are provided by the manufacturer as package units. OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study

# 17.0 - 2.5 MGD VIE PLANT EQUIPMENT LIST AND DRAWING INDEX

Sheet 1 of 4

### 17.1.1 - EQUIPMENT LEST FOR 2.5 MGD VIE PLANT - SMOOTH TUBES

tubes, 24' long. First Effect Evaporator - 9' x 40' high with 1-3/4" 0.D. .049" wall smooth 90-10 Cu-Ni  $51^4$  fX - <u>Preheater</u> - 2-pass with 3/4"0.D. .049" wall smooth 90-10 Cu - Ni tubes, 22' long. 면 디

M-1 Module - Containing 12 preheaters as follows:

long	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
tubes											
smooth	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
preheater with 3/4" 0.D035" wall	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
2-pass preheater with	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
Η	30.3 HX										

and 13 effect tube bundles as follows:

long										
		ф ф	ф	do.	do.	do.	do.	ф	ф	٠ و
tubes t										
smooth 90-10 Cu - Ni	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
2" 0.D049" wall	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
	do.							dc.		
Second Eff	Third	Fourth	Fifth	Sixth	Seventh	Eighth	Ninth	E-10 Tenth	Eleventh	E-12 Twelfth
Щ С	E-77	元 元 元	표 근	E-C	E- 7	E-8	田 シー	E-10	E-17	E-12

# 17.1.1 - EQUIPMENT LIST FOR 2.5 MGD VIE PLANT - SWOOTH TUBES (Cont'd.)

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study Sheet 2 of 4

### M-1 Module (Cont'd.)

2-1/2" 0.D. x .049 wall smooth 90-10 Cu · Ni tubes 24' long tubes 24' long E-13 Thirteenth Effect tube bundle with 2" 0.D. .049" wall smooth 90-10 Cu-Ni E-14 Fourteenth do. 2-1/2" 0.D. x .049 wall smooth 90-10 Cu

and --

tubes E-15 Heat Rejection Condenser with 3/4" 0.D. 0.049" wall smooth 90-10 Cu-Ni

Heat Transfer Surfaces - Preheaters 302-314 HX = 47,266 Sq. Ft. Evaporator Effects E-1 thru E-14 = 162,046 Sq. Ft. Heat Rejection 3,540 Sq. Ft.

Tube Sheets and Channels - 90-10 Cu - Ni or Cu - Ni - Clad, Min. thickness of cladding - 20% of

Water Boxes, return heads - 90-10 Cu - Ni or fiberglass

### E-16 Product Cooler

2 shell and tube exchangers -  $2^{4}$ \*\* 0.D. x  $3^{1}$ -4" long with 3/4" 0.D. 0.049" wall smooth 90-10 Cu - Ni tubes 26' long

### V-1 Deserator-Decarbonator

13'-6" I.D. x 26'-0" T-T Vert. w/Dow "Mas Pac" packing

Pumps and Drivers				Head		Elec. Motor
		Type		ft.	Material	-•
*P-1A & B	Raw Seawater Feed	Horiz.	~	55	All 316 S.S.	
P-2A & B	Deaer. Seawater Feed	Vert-Can	n 3200	261	do.	1780
P-12	Falling Film Feed	Horiz.		35	do.	
P-13	do.	do.		35	do.	
P-123	do. (spare)	do.	٠.	35	do.	
P-14	do.	do.		35	do.	
P-15	do.	do.	•	35	do.	
P-145	do. (spare)	do.	• •	35	do.	
P-16	do.	do.	•	35	do.	

\*\*Dimensions are approximate. \*Not included in price of plant in reference VTE proposal

17.1.1 - EQUIPMENT LIST FOR 2.5 MGD VTE PLANT - SMOOTH TUBES (Cont'd.)

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study

Sheet 3 of 4

Pumps and Drivers (Cont'd.)

Elec. Motor	Speed, RPM	1150	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	1750	do.	1150	1750	do.	do.	
E	·	A11 316 S.S.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	C.I. case, 316 S.S. trim	do.	do.	316 S.S. trim	do.	A11 316 S.S.	C.I. case, 316 S.S. trim	Monel	All 316 S.S.	
Head	ft.	35	35	35	35	35	35	35	35	35	35	35	35	55	35	12	12	85	55	250	99	100	100	200	
	GPM	1800	2100	1800	1800	1800	2100	2100	2100	1800	1800	1800	1800	1100	1800	120	120	2080	0.25	0.1	8250	215	100	88	
	Type	Horiz.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	Horiz. Prop.	do.	Vert.	Vert.	Horiz.	do.	
		m Feed	$\overline{}$			do. (spare)			(spare)			(spare)	do.	Blowdown	Spare for 24 & 25	Heat Reject Distillate	HR Distillate (spare)	Product	Acid	Caustic	Seawater Intake	Condensate Return	Chlorine	Screen Cleaning	
		P-17	P-167	P-18	P-19	P-189	P-20	P-21	P-201	P-22	P-23	P-223						В	M	B	М	Щ	Д	Щ	

Instrument Air Compressor - 100 scfm at 80 psig nonlubricated, motor-driven w/aftercooler and receiver tank \*C-2 \*C-1

\*Not included in price of plant in reference VTE proposal \*\*Added by Fluor

Service Air Compressor - 100 scfm at 100 psig, motor-driven  $\rm w/aftercooler$  and receiver tank

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study

Sheet 4 of 4

### Tanks

TK-1 Acid Storage - 8'Ø x 14'-0" T-T TK-2 Caustic Storage - 4'Ø x 11' T-T Condensate Drum - 2'Ø x 9'-3" Vert. TK-4 Caustic Mix Tank with Agitator - Open Tank 5'Ø x 5'high

### Miscellaneous Equipment

Each set contains: 2-Stage Ejector and Intercondenser, and installed spare. Щ Set iA

One Barometric Precondenser and two Barometric Intercondensers - Fiberglass or C.I. or rubber-lined. Ejectors have S.S. nozzles and C.S. chest.

\*\*Instrument Air Drier - 100 scfm with prefilter and afterfilter

\*\*Trash Bar w/traveling rake

\*\*Traveling Screen - 3/8" S.S. mesh

\*\*Chlorination System - Ejector, metering system, pressure reducing valve.

\*\*Communication System

### Buildings and Structures

Control House and Control Board Motor Control Center Seawater Intake Structure

### Warehouse Spares

None

\*\*Added by Fluor

17.0 - 2.5 MGD VTE PLANT EQUIPMENT LIST
AND DRAWING INDEX
MSF

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study

Sheet 1 of 4

### 17 1.2 - EQUIPMENT LIST FOR 2.5 MGD VTE PLANT - ENHANCED TUBES

First Effect Evaporator - 8'\( \precedef{x} \) x 30 ft. high with 2" OD. 0.049" wall double-fluted 90-10 Cu-Ni tubes, 14' long E-I

lows:	
as foll	
preheaters	
57	
Containing	
ı	
Module	
M-1	

	long	ф	ф •	<b>do.</b>	qo.	do.	do.	do.	<b>.</b> ob	qo.	qo.	qo.	do.
,	tubes								12,				
	wall spirally grooved 90-10 Cu -Ni	•op	do.	· op	do.	do.	.ob	do.	do.	do.	do.	· op	do.
-	preheater with 1" 0.D. x 0.042" wall spirally grooved	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
•	. 2-pass preheater	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.
									310 HX				314 HX

## and 13 effect tube bundles as follows:

sapes						do.		
Second Effect tube bundle with 2" 0.D. 0.049" wall double-fluted 90-10 Cu -Ni t	.ob	do.	do.	do.	do.	do.	do.	do.
le with 2" 0.D. 0.049" wall	do.	do.	do.	do.	do.	do.	do.	do.
fect tube bund	do.	do.	do.	do.	do.	do.	do.	do.
E-2 Second Ef	E-3 Third	E-4 Fourth	E-5 Fifth	E-6 Sixth	E-7 Seventh	E-8 Eighth	E-9 Ninth	E-10 Tenth

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study

Sheet 2 of 4

### M-1 Module (Cont'd.)

tubes 14' long E-11 Eleventh Effect tube bundle with 2" 0.D. 0.049" wall double-fluted 90-10 Cu -Ni tubes 14' long 5" 0.D. x 0.049" wall double-fluted 90-10 Cu -Ni ф ф ф ф E-12 Twelfth E-13 Thirteenth E-14 Fourteenth

and --

tubes E-15 Heat Rejection Condenser with 1" 0.D. x 0.049" wall enhanced surface 90-10 Cu -Ni

- Preheaters 302-314 HX = 71,155 Sq. Ft. Evaporator Effects E-1 thru E-14=78,598 Sq. Ft. Heat Rejection = 2,818 Sq. Ft. Heat Transfer Surfaces

Tube Sheets and Channels - 90-10 Cu.-Ni or Cu -Ni - clad, min. thickness of cladding - 20% of total. Water Boxes, return heads - 90-10 Cu - Ni or fiberglass

### E-16 Product Cooler

l shell and tube exchanger - 36"\*\* 0.D. x 27'-6" long with 1" 0.D..049" wall enhanced surface 90-10 Cu -Ni tubes 20' long.

### V-1 Deaerator-Decarbonator

13'-6" I.D. x 26'-0" T-T Vert. w/Dow "Mas Pac" packing.

Elec. Motor Speed, RPM		1780				
Material S	All 316 S.S.	· op	do.	do.	do.	do.
Head ft.	55	380	45-55	45-55	45-55	45-55
GPM	-	3200				
Type	Horiz.	Vert-Can	Horiz.	do.	do.	do.
	Non-deaer. Seawater Feed	Deaer. Seawater Feed	Falling Film Feed	do.	do. (spare)	do.
Pumps and Drivers	*P-1A & B	P-2A & B			P-123	P-14

\*Not included in price of plant in reference VTE proposal \*\*Dir

sal \*\*Dimensions are approximate.

17.1.2 - EQUIPMENT LIST FOR 2.5 MGD VTE PLANT - ENHANCED TUBES (Cont'd.)

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study Sheet 3 of 4

Pumps and Drivers (Cont'd.)

Elec. Motor Speed, RPM	1150	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	SS trim do.	do.	do.	1750	do.	1150	ss 1750	do.	do.
Material	A11 316 S.S.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	C.I. case, 316 SS	do.	do.	316 s.s.	do.		case, 316	All monel	A11 316 S.S.
Head ft.	45	45	45	45-55	45	45-55	45-55	45-55	45	45	45	45-55	45-55	45-55	45-55	55	45-55	12	12	8 55	55	375	99	100	100	200
GPM	2500	2500	2100	1800	2100	1800	1800	1800	2100	2100	2100	1800	1800	1800	1800	1100	1800	120	120	2080		0.1	8250	215	100	88
Type	Horiz.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	do.	te do.	(spare) do.	do.	Horiz. Prop	do.	Vert.	Vert.	Horiz.	do.
	Falling Film Feed	do. (spare)		do.	do. (spare)		do.	do. (spare)	do.	do.	do. (spare)		do.	do. (spare)	do.	Blowdown	Spare for 24 & 25	Heat Reject Distillate	Heat Reject Distil.	Product	Acid	Caustic	Seawater Intake	Condensate Return	Chlorine	Screen Cleaning
	P-15	P-145	P-16	P-17	P-167	P-18	P-19	P-189	P-20	P-21	P-201	P-22	P-23	P-223	P-24					ഷ മ	න መ	ഷ മ	& B	ж Ш		**P-55A & B

\*\*Added by Fluor

OSW Contract 14-01-0001-1830 Fluor Contract 4334 MSF and VTE Comparative Study Sheet 4 of 4

### Compressors

Instrument Air Compressor - 100 scfm - 80 psig nonlubricated, motor-driven w/aftercooler and receiver tank. \*C-7 \*C-2

Service Air Compressor - 100 scfm - 100 psig, motor-driven w/aftercooler and receiver tank.

### Tanks

Saustic Mix Tank with Agi**t**ator - Open Tank 5'ø x 5' high - 8'Ø x 14'-0" T-T - 4'Ø x 11'-0" T-T - 2'Ø x 9'-3" Vertical Caustic Storage Condensate Drum Acid Storage TK-1 TK-2 TK-3 \*\*TK-4

### Miscellaneous Equipment

Each set contains: 2-Stage Ejector and Intercondenser, and installed spare. Set lA & B

One Barometric Precondenser and two Barometric Intercondensers - Fiberglass or C.I. or Ejectors have S.S. nozzles and C.S. chest. rubber-lined.

\*\*Instrument Air Drier - 100 scfm with prefilter and afterfilter.

\*\*Trash Bar w/traveling rake

\*\*Traveling Screen - 3/8" S.S. mesh

\*\*Chlorination System - Ejector, metering system, pressure reducing valve.

\*\*Communications System

### Buildings and Structures

Control House and Control Board Seawater Intake Structure Motor Control Center

### Warehouse Spares

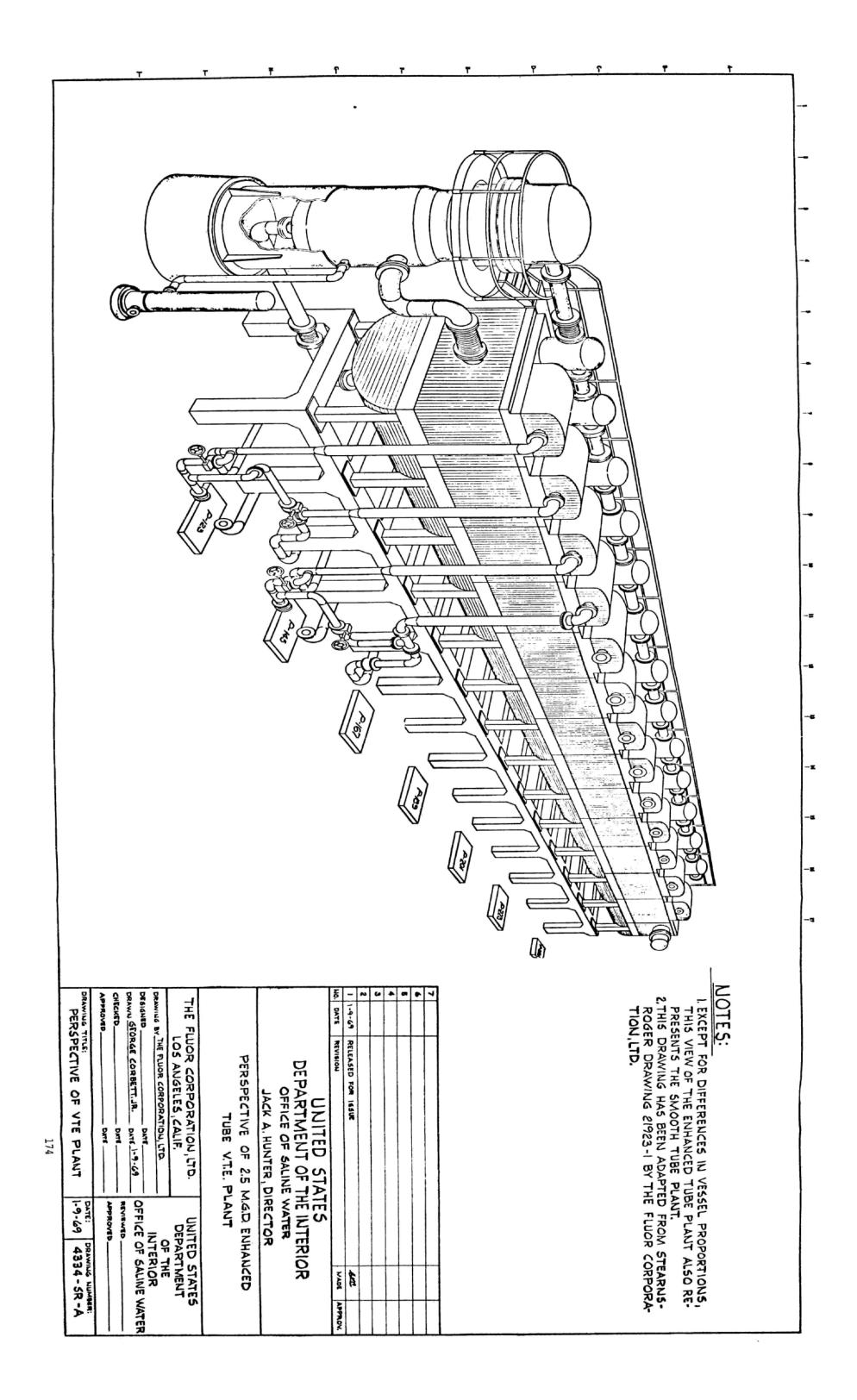
None

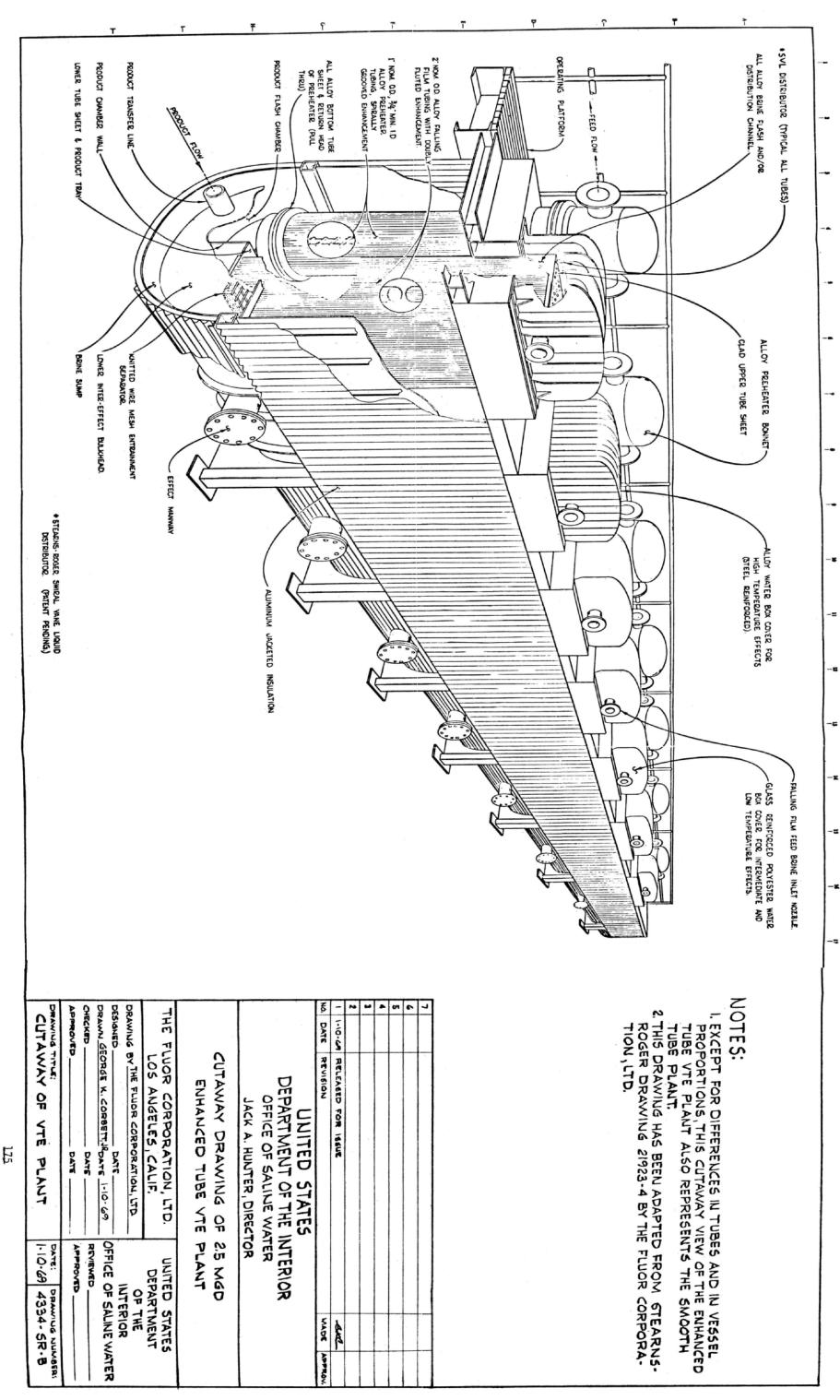
\*Not included in price of plant in reference VTE proposal \*\*Added by Fluor

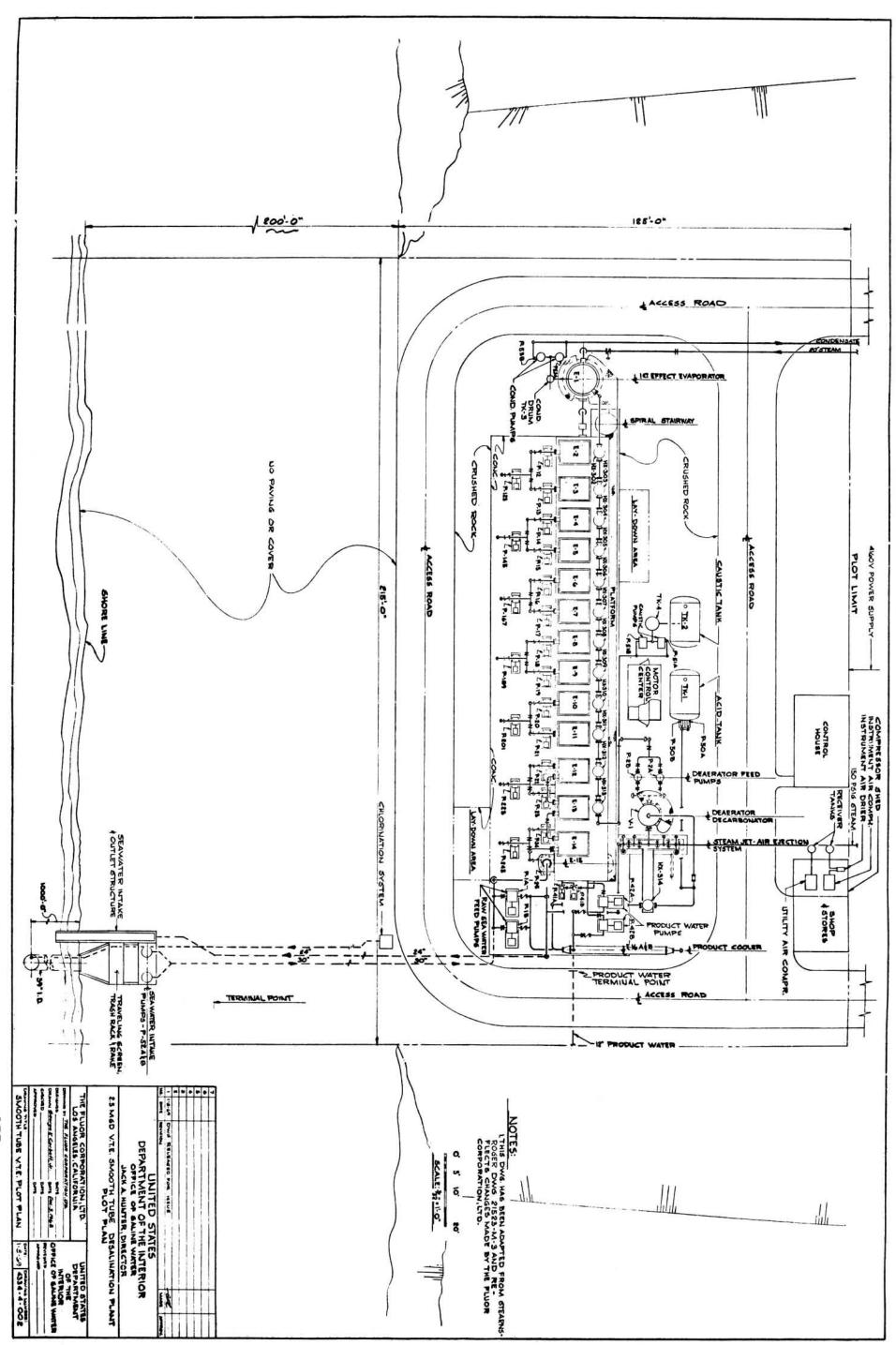
#### 17.0 - 2.5 MGD VTE PLANT EQUIPMENT LIST AND DRAWING INDEX

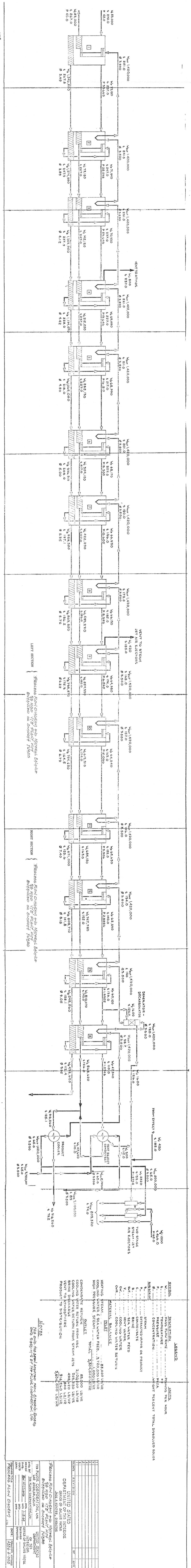
#### 17.2 Drawing Index

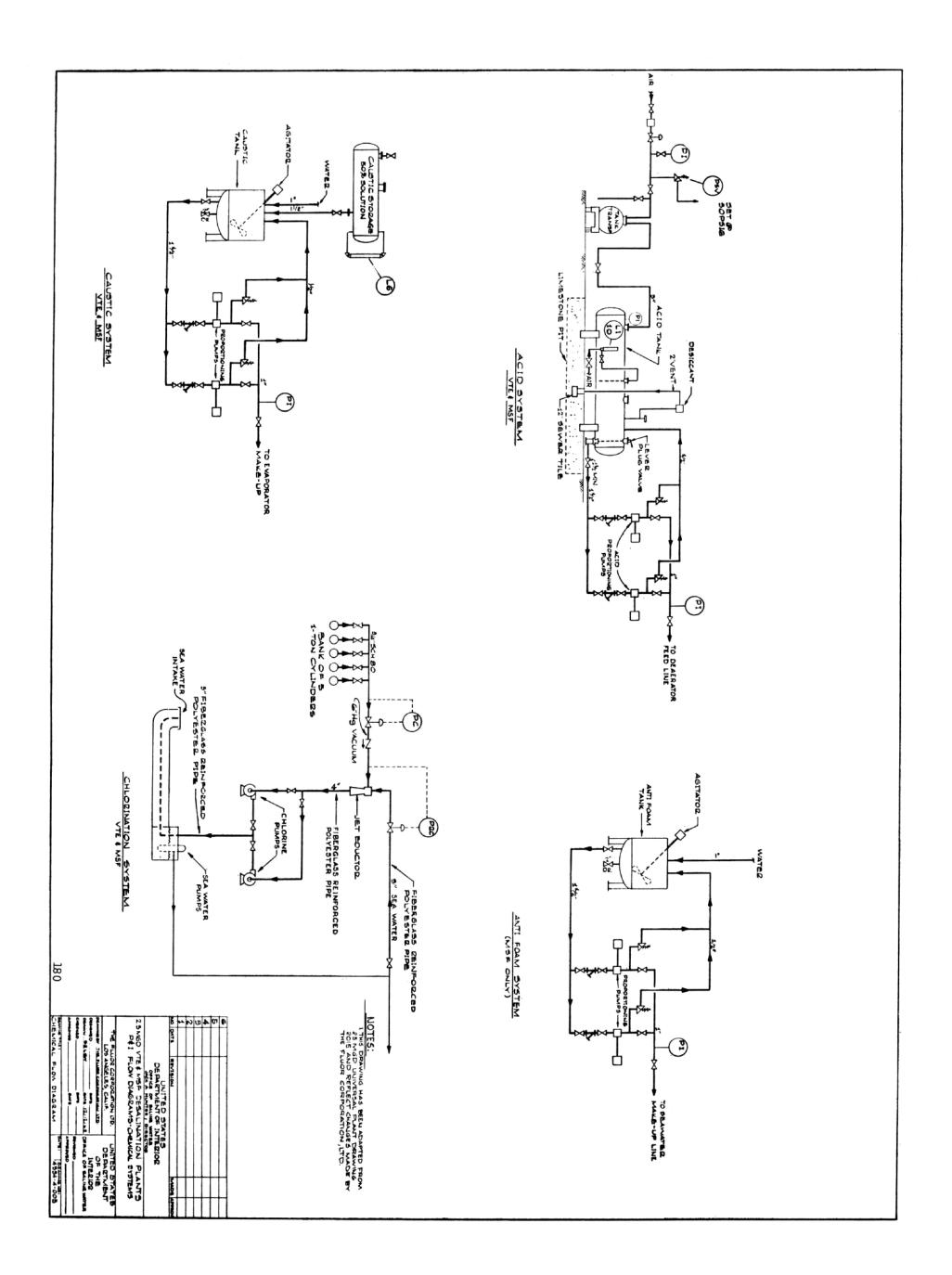
Drawing Number	<u>Title</u>
4334-SR-A	Perspective of VTE Plant
SR-B	Cutaway of VTE Plant
4-001	Enhanced VTE Plot Plan
4-002	Smooth Tube VTE Plot Plan
4-003	Process Flow Diagram
4-004	Mechanical Flow Diagram
4-005	Chemical Flow Diagram
4-006	Electrical One-Line Diagram

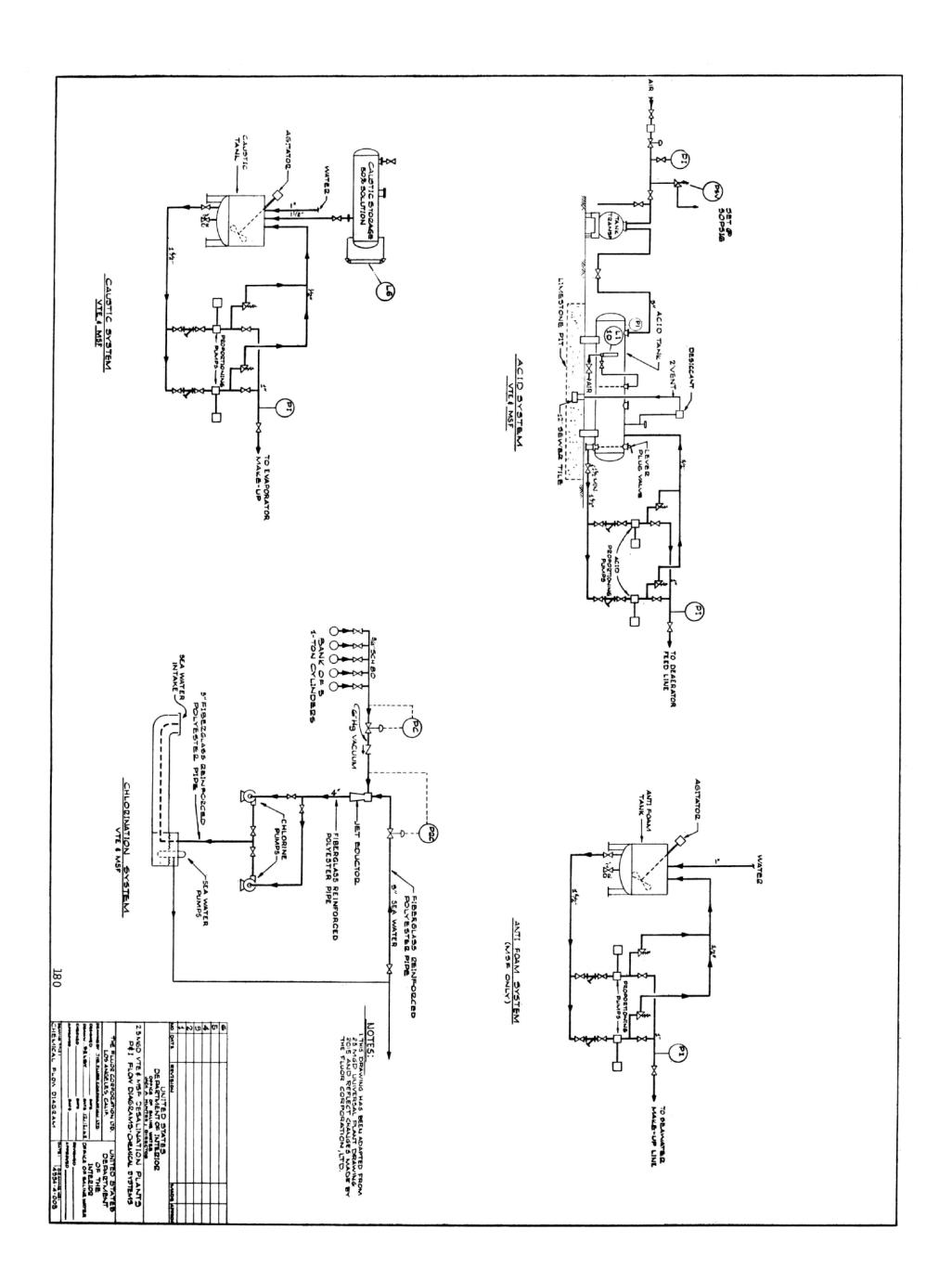


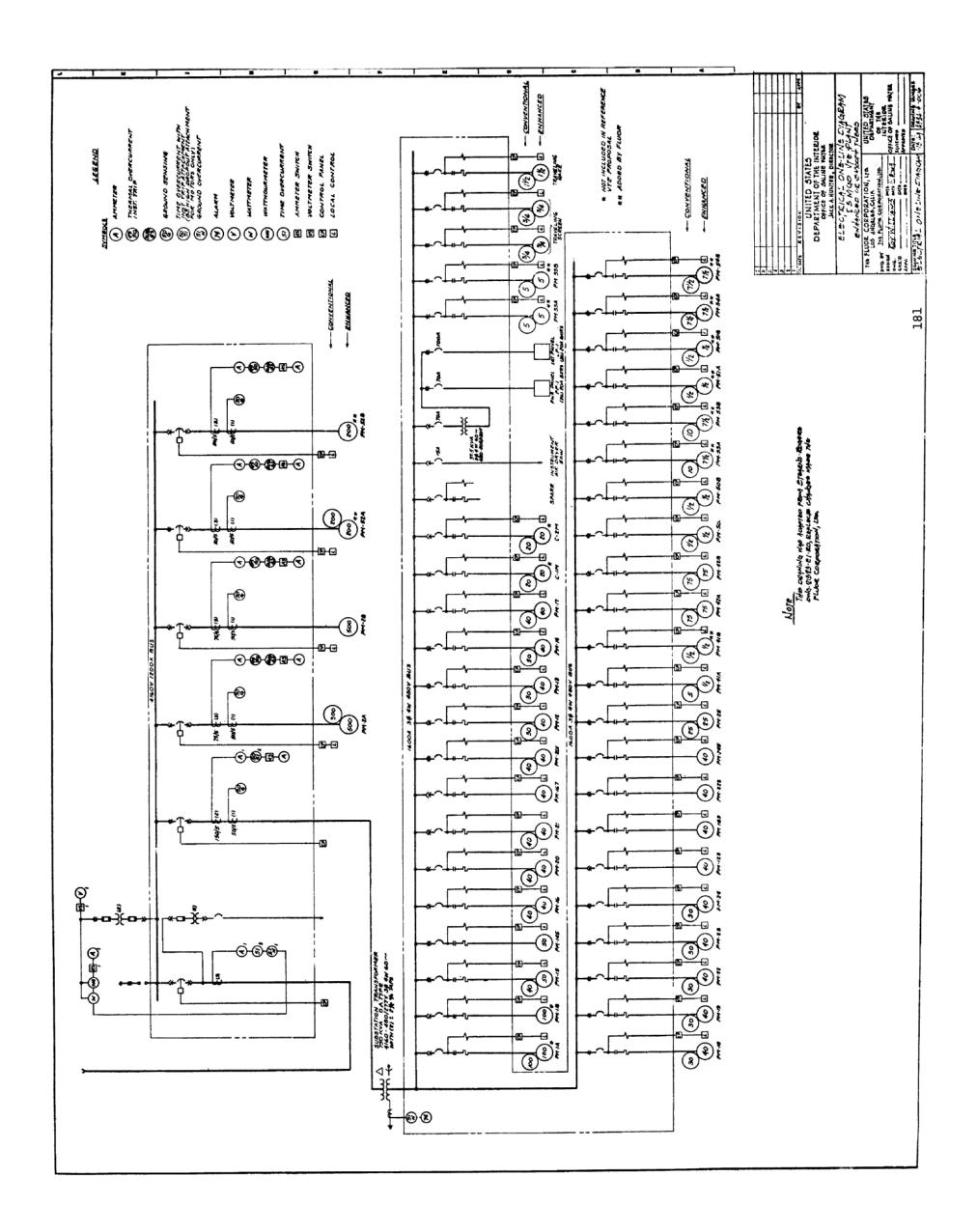












18.0 - 2.5 mgd msf Plant equipment list and draming index

OSW Contract 14-01-0001-1850 Fluor Contract 4334 MSF and VTE Comparative Study Sheet 1 of 3

#### 18.1 - EQUIPMENT LIST FOR FOR PLANT - SMOOTH T

#### Flash Evaporator Modules

Heat Rejection Module - 8'-6" x 12' high x 63' long - 3 stages - Tubes 3/4" 0.D. x .035" wall 90-10 Cu-Ni M-5-7 M-2 M-7

Tube Sheets - 90-10 Cu-Ni or Clad Water Box - 90-10 Cu-Ni above 180 F

Water Box - Carbon steel below 180 F

- 46" dia. x 16'-6" T-T, tubes 3/4" 0.D. x .035" wall 90-10 Cu -Ni, - 15'-6" long Brine Heater E-1

- 15'-4" dia. x 26'-0" T-T vertical with Dow "Mas Pac" packing Deaerator-Decarbonator TK-1

P-1A & B Brine Rec P-2A Blowdown) *P-2B Blowdown)				неад		Tec. Motor
		Type	GPM	ft	Material	Speed, RPM
ш ш -	Brine Recycle (50% Cap.)	Vert.	9069	203	All 316 S.S.	1150
' -	мт.) (т	Vert.	2000	61	do.	1150
٦	·+ ·	Horiz.	2080	85	C.I. case, 316 S.S. trim	
. & B Acid		Horiz. Prop.	0.25	21	316 8.8.	1750
. & B Anti-foam	oam	Do.	0.1	100	316 s.s.	1750
& B Caustic	O	Do.	0.1	155	316 s.s.	1750

\*Added by Fluor

<b>OSW</b> Contract 14-01-0001-1830	Fluor Contract 4334	MSF and VTE Comparative Study
18.1 - EQUIPMENT LIST FOR 2.5 MGD MSF PLANT - SMOOTH TUBES (Cont'd.)		

Sheet 2 of 3

#### Pumps and Drivers (Cont'd.)

				Head	Н	Mec. Motor
		Type	GPM	ft.	Material	Speed, RPM
*P-8A & B	Seawater Intake	Vert.	12,000	99	66 All 316 S.S.	870
*P-9A & B	Condensate Return	Vert.	208	100	C.I. Case, 316 S.S. trim	1750
*P-lOA & B Chlorine	Chlorine	Horiz.	100	100	All monel	1750
*P-11A & B	*P-11A & B Screen Cleaning	Horiz.	88	200	A11 316 S.S.	1750
Compressors						
C-1	Instrument Air Compressor Motor-driven w/aftercooler E-3 and receiver Tk-8	E-3 and	receiver Tk-8			

Motor-driven w/aftercooler E-3 and receiver TK-8 100 scfm at 80 psig - nonlubricated	Service Air Comp. Motor-driven w/aftercooler E-4 and receiver Tk-9 100 scfm at 100 psig
i >	C-5

#### Tanks

- Open Tank $5' \beta \times 5'$ high	- 4' 0.D. x 10' T.T. horizontal	- 9' 0.D. x $14$ ' T.T. horizontal	2 - 0 open Tank $5'$ ø x $5'$ high
Anti-foam Tank with Agitator A-1	Caustic Storage Tank	Acid Storage	Caustic Mixing Tank with Agitator A-2 - Open Tank $5'0 \times 5'$ high
TK-4	TK-5	玩-6	TK-7

#### Miscellaneous Equipment

EJ-1

One two-stage steam jet air ejector with inter-and aftercondensers

Hogging Ejector EJ-2

\*Added by Fluor

Fluor Contract 4554 **OSW** Contract 14-01-0001-1830 18.1 - EQUIPMENT LIST FOR 2.5 MGD MSF PLANT - SMOOTH TUBES (Cont.d.)

Fluor Contract 4,294 MSF and VTE Comparative Study

Sheet 3 of 3

Miscellaneous Equipment (Cont'd.)

E-2 Vent Condenser

Instrument Air Drier - 100 scfm, with prefilter and afterfilter TK-3

\*Trash Bar w/traveling rake

\*Traveling Screen - 1/2" S.S. mesh

\*Chlorination System - Ejector, metering system, pressure reducing valve.

\*Communication System

### Buildings and Structures

Control Building and Control Board Switchgear Building Seawater Intake Structure

#### Warehouse Spares

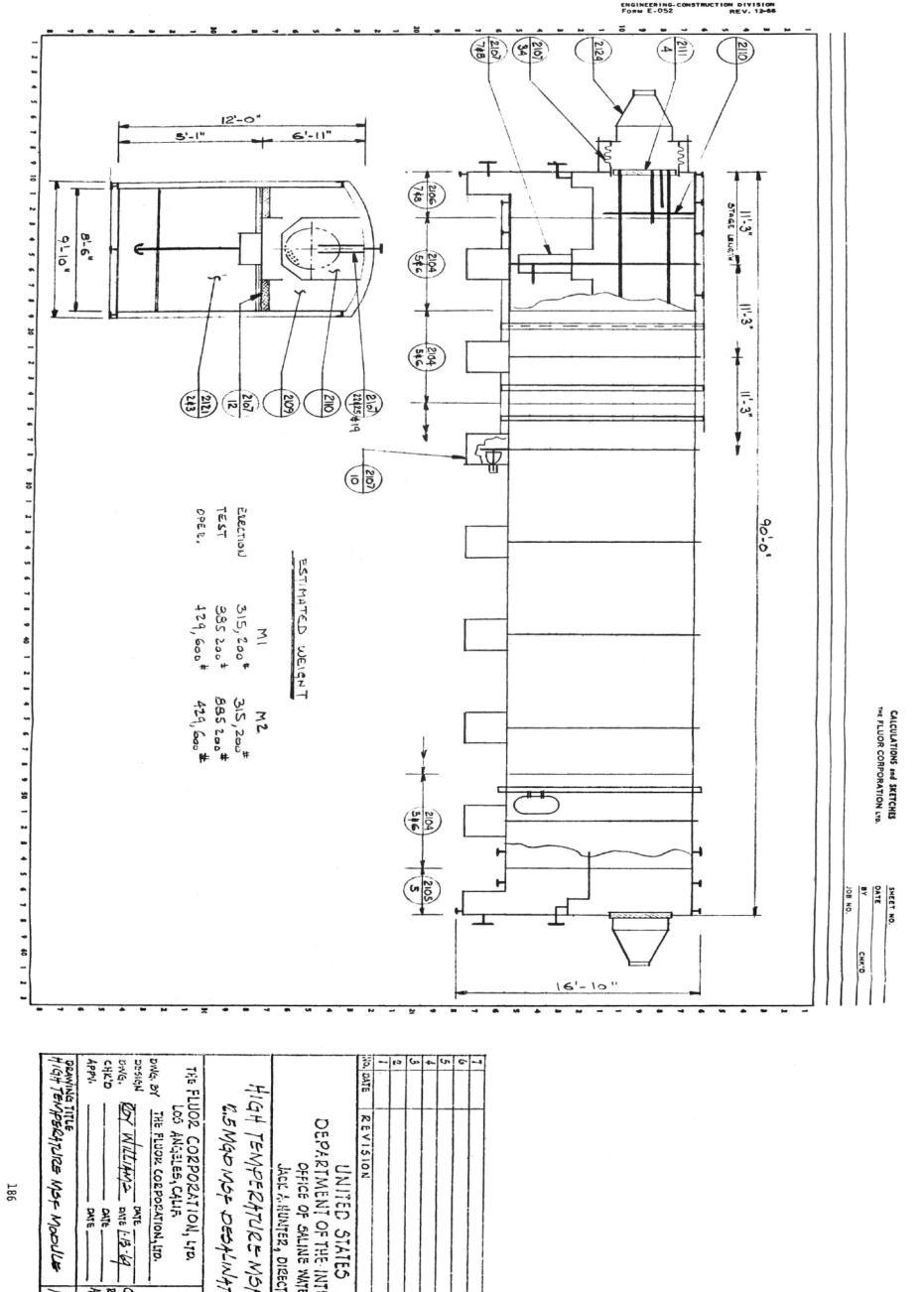
None

#### 18.0 - 2.5 MGD MSF PLANT EQUIPMENT LIST AND DRAWING INDEX

#### 18.2 <u>Drawing Index</u>

Drawing Number	<u>Title</u>
4334-FC-A	High Temperature MSF Module
FC-B	Low Temperature MSF Module
FC-C	Heat Rejection MSF Module
4-011	MSF Plot Plan
4-012	Process Flow Diagram
4-013	Mechanical Flow Diagram
4-005	Chemical Systems
4-014	MSF Electrical One-Line Diagram





NO. DATE REVISION DEPARTMENT OF THE INTERIOR ঘ APPV.

HIGH TEMPERATURE MOF MODULE

JACK A HUNTER, DIRECTOR

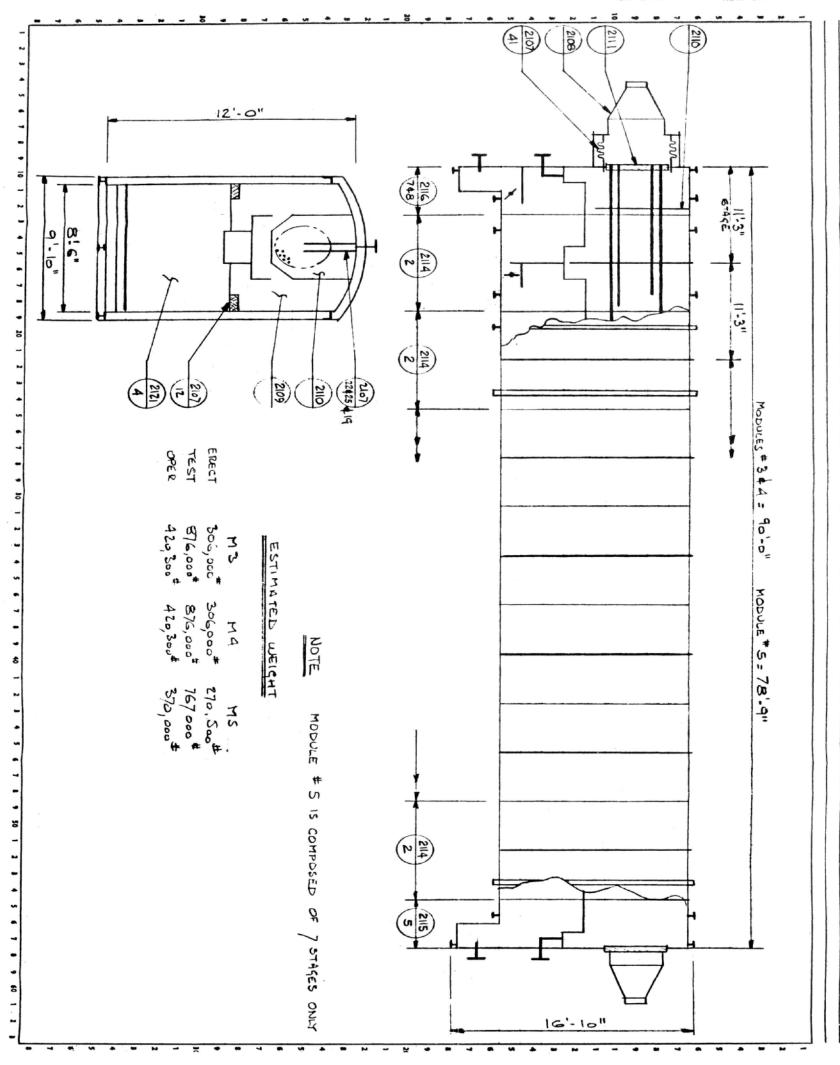
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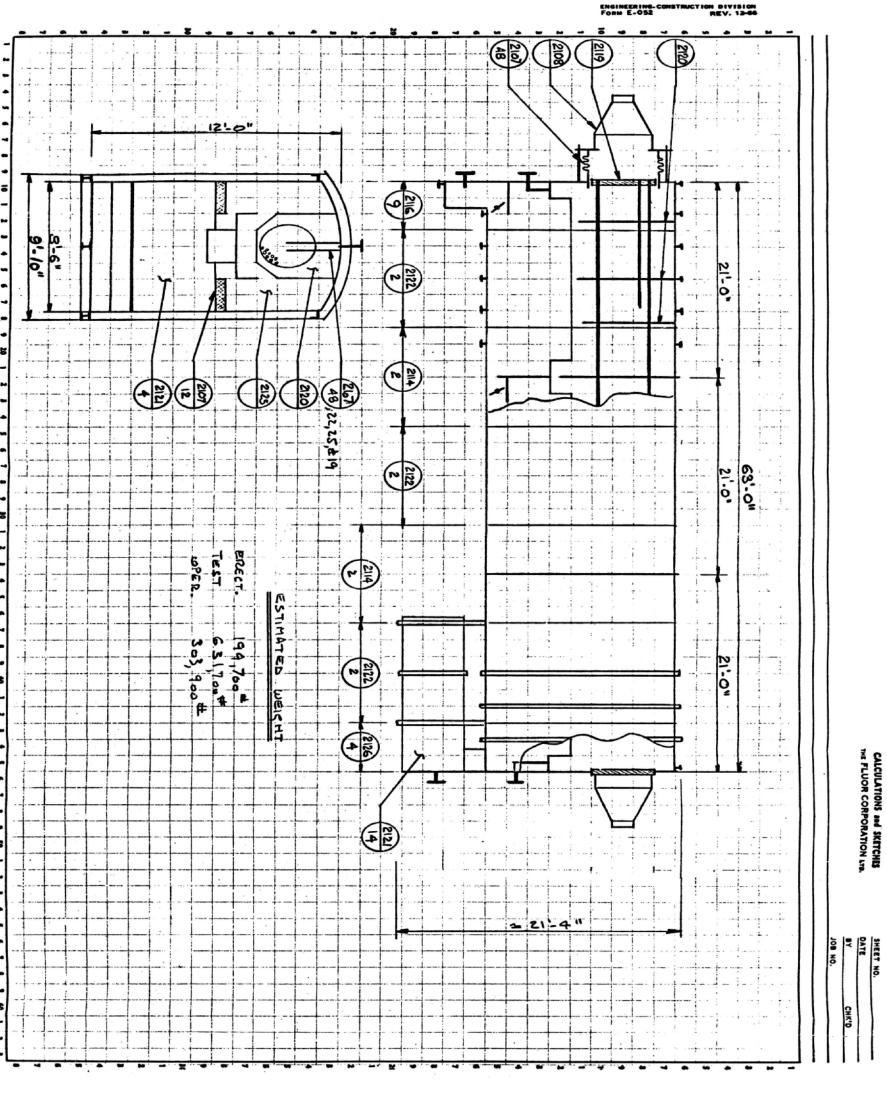




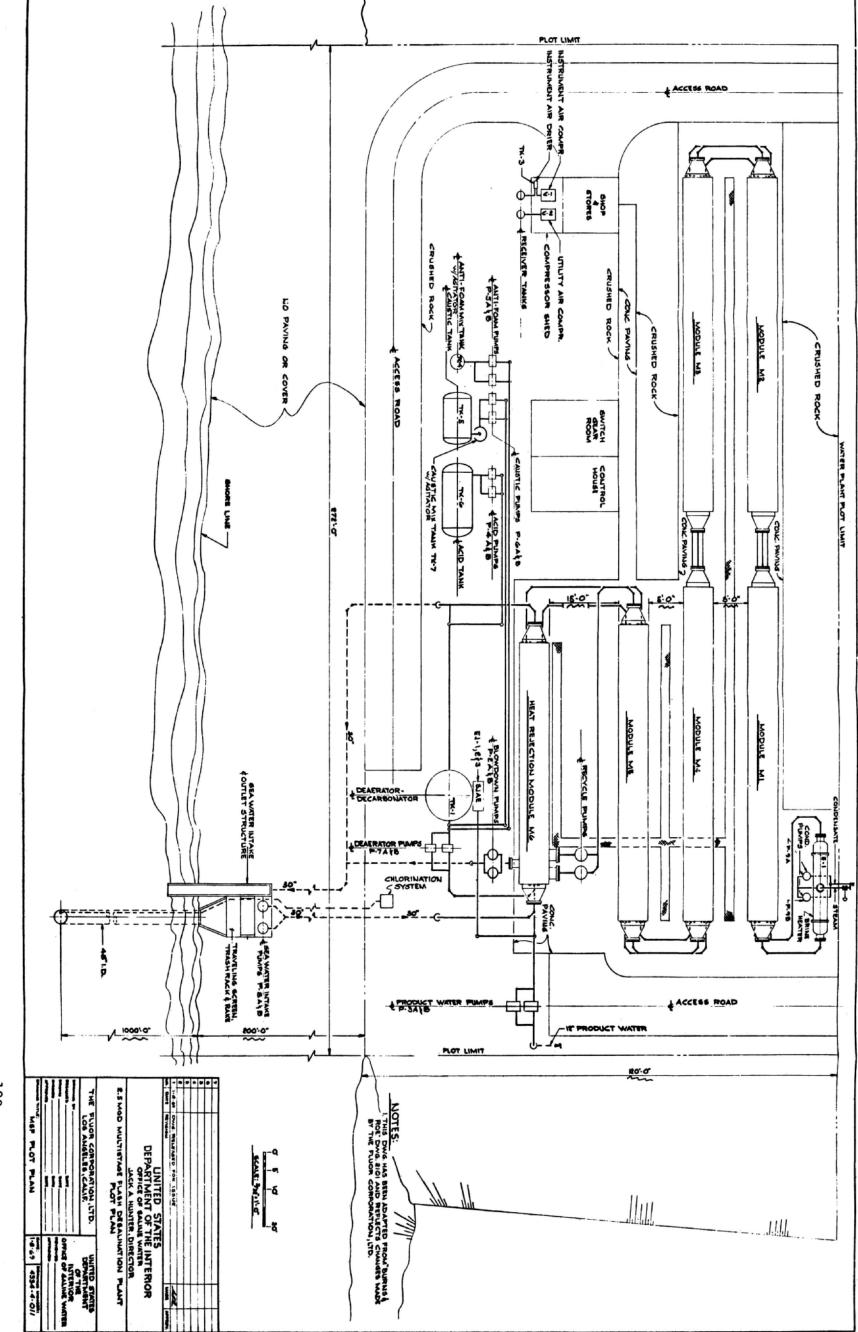
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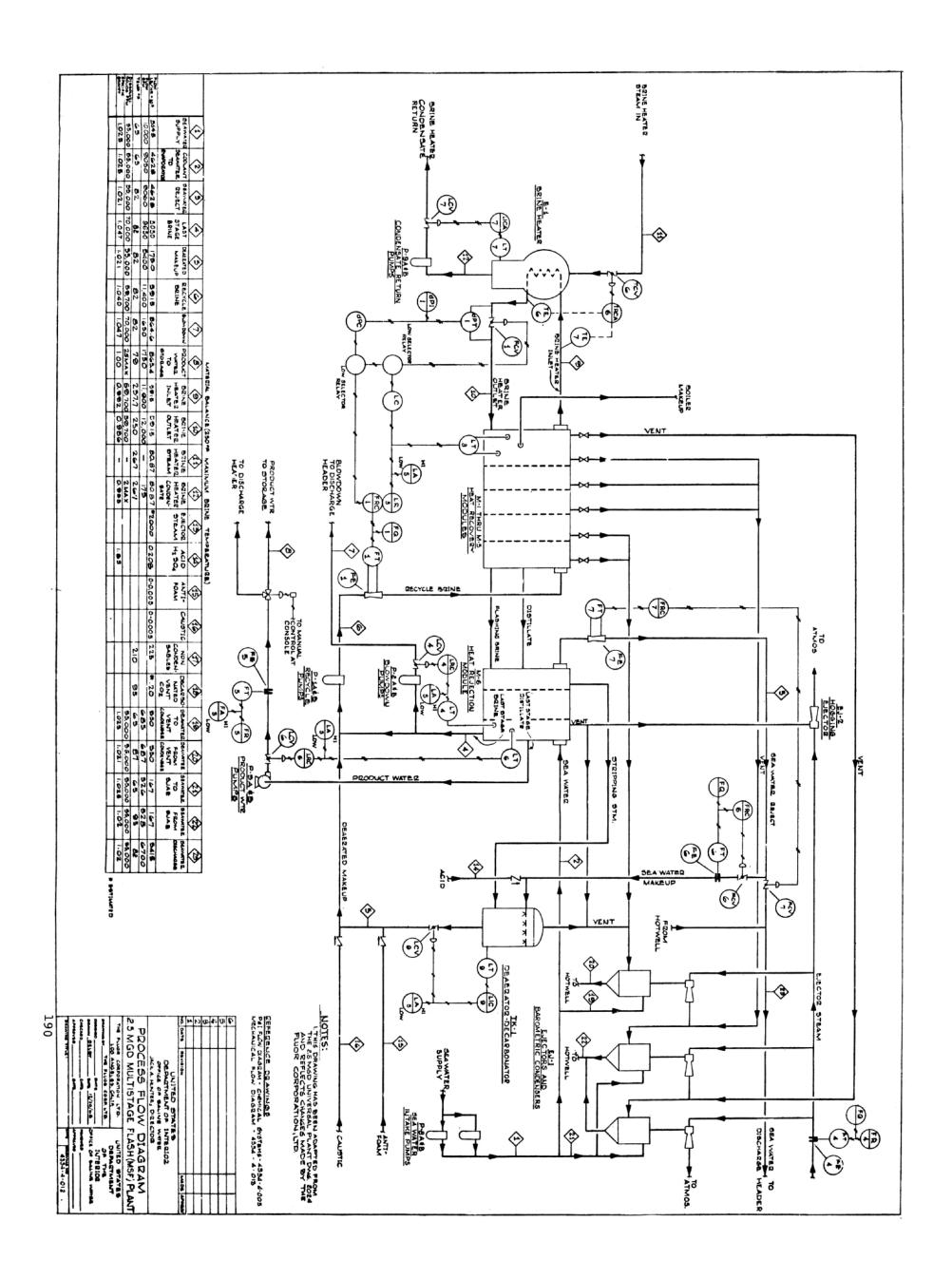
CALCULATIONS and SKETCHES

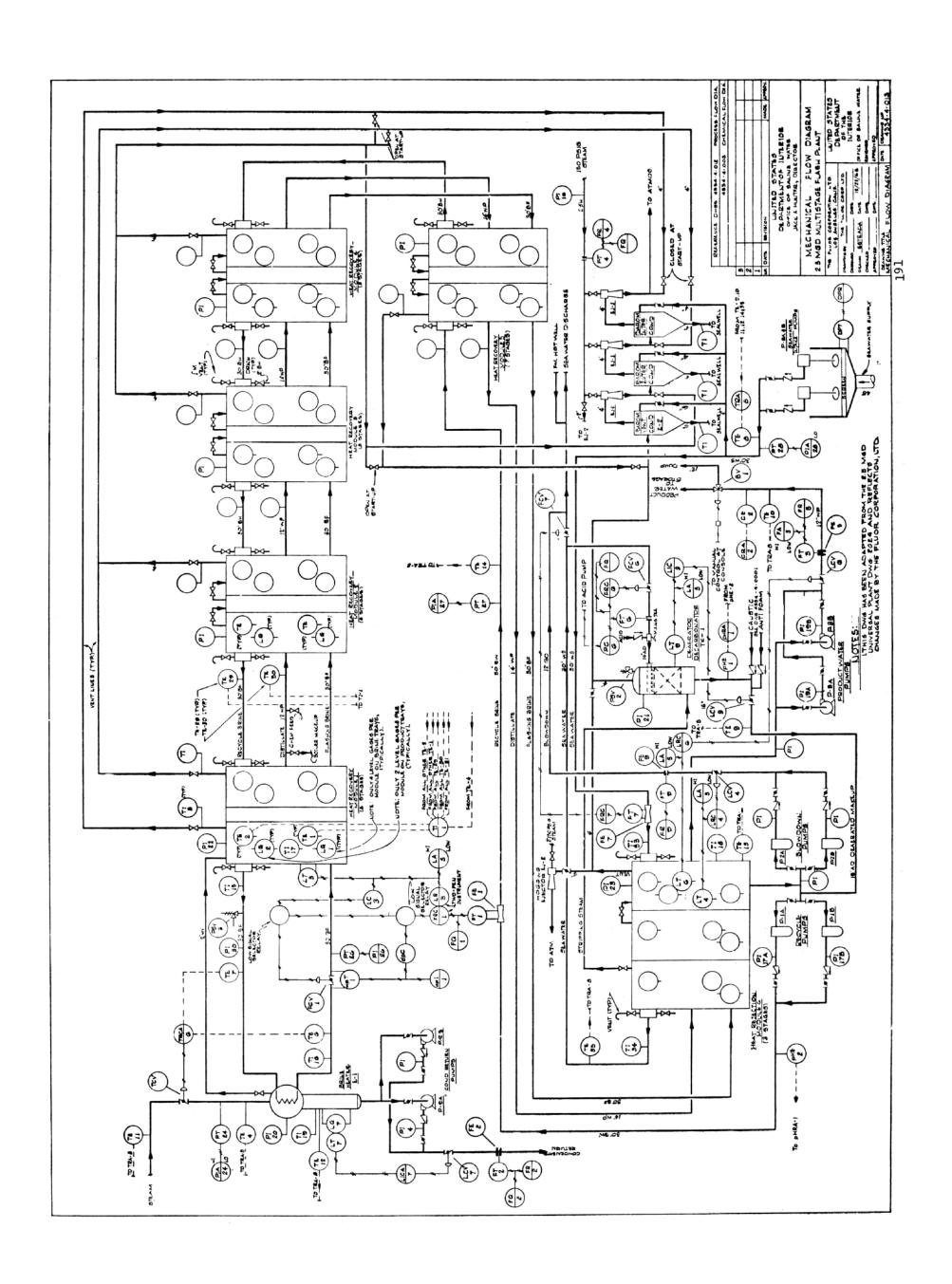
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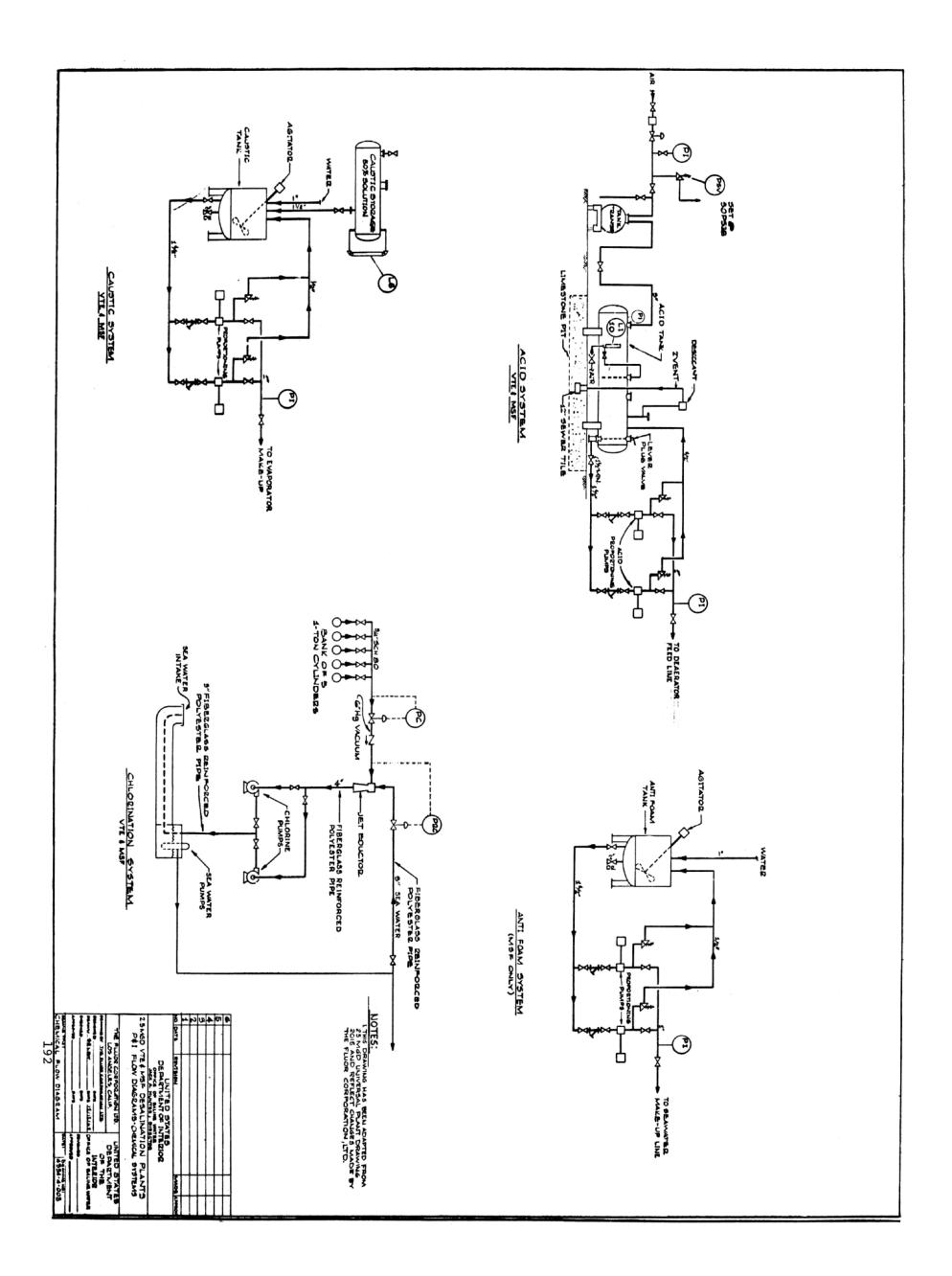


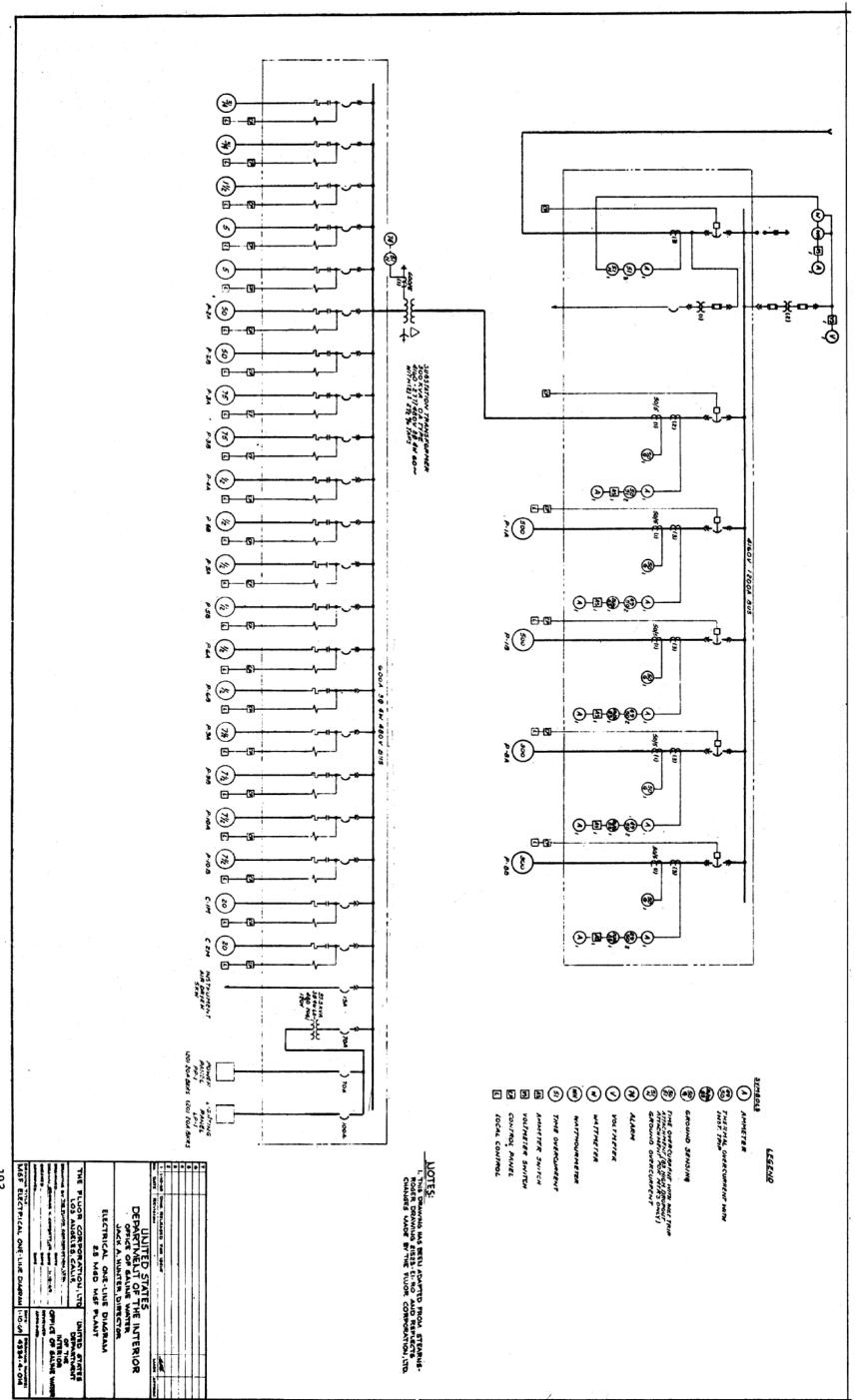
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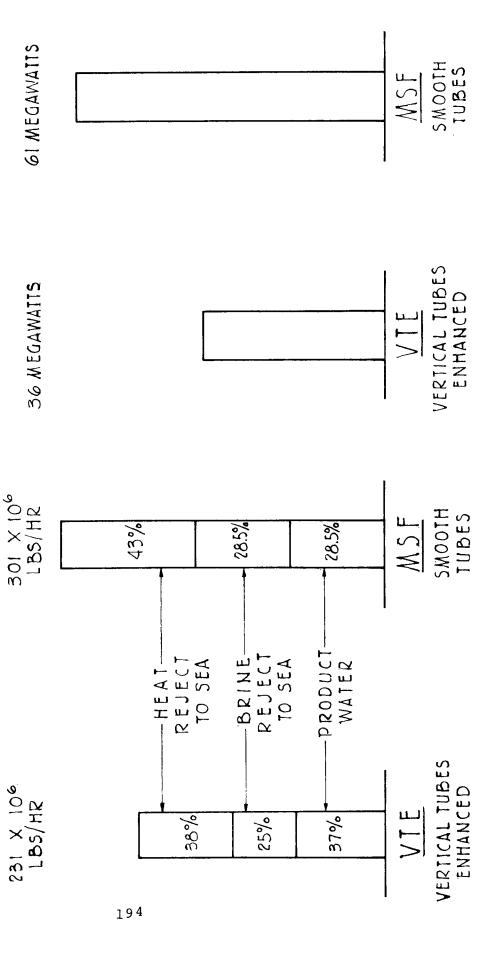




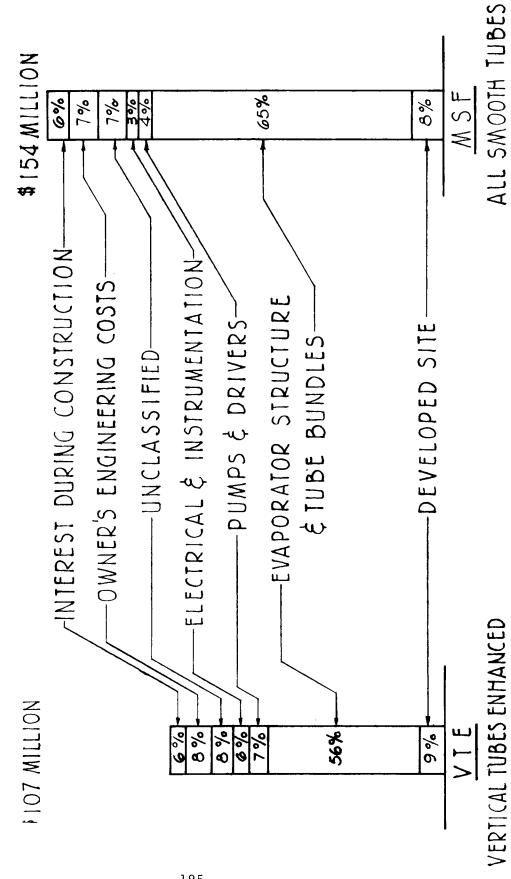
# LARGE (250MGD) PLANTS

# SEAWATER INTAKE FLOWS

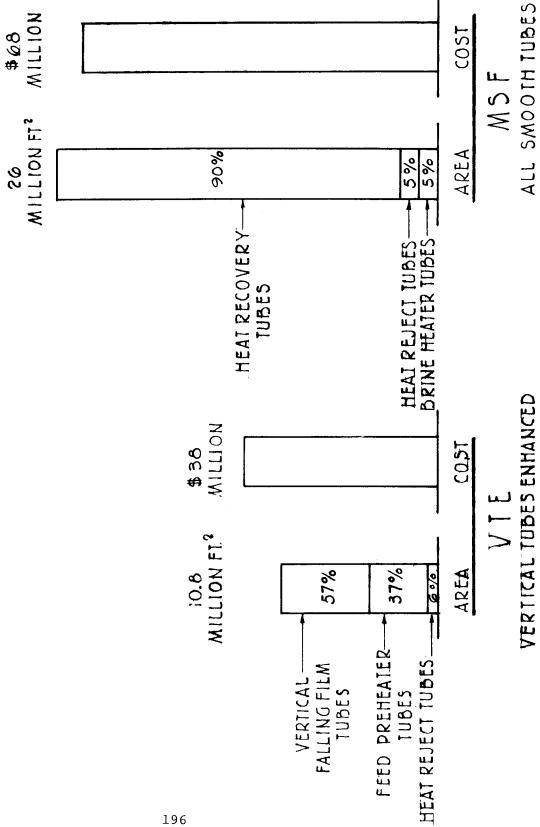
# PUMPING POWER REQUIREMENTS



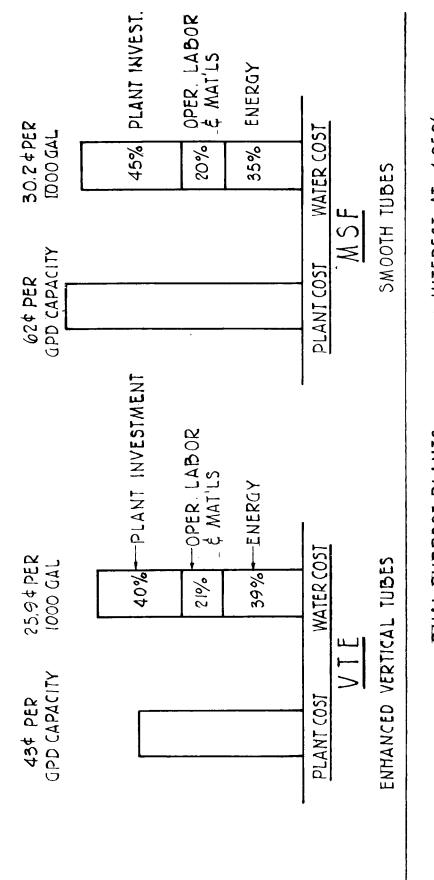
### LARGE (250MGD) PLANTS CAPITAL COSTS



# HEAT TRANSFER AREAS AND COSTS LARGE (250MGD) PLANTS

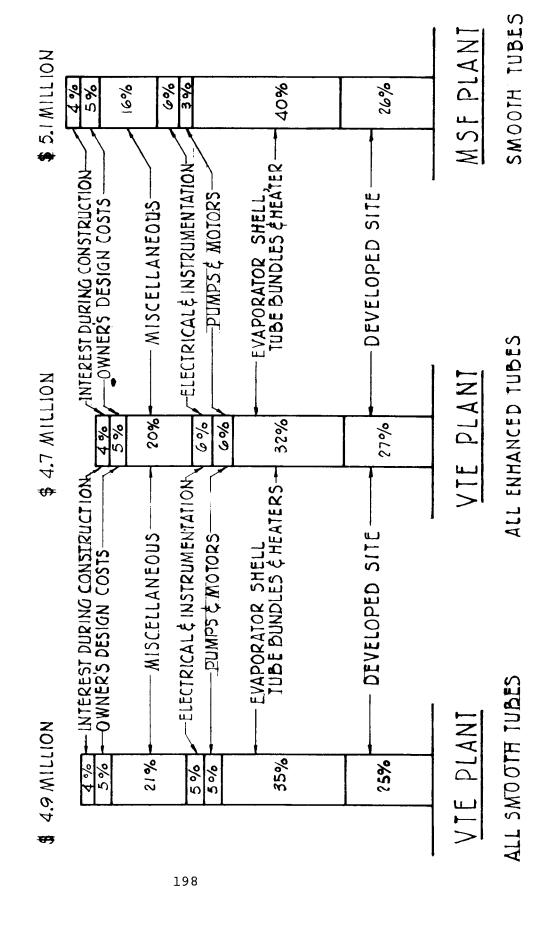


### APITAL AND WATER COSTS LARGE (250MGD) PLANTS CAPITAL



- DUAL PURPOSE PLANTS STEAM AT 11.2-12¢/10°BTU. 30 YEAR PLANT LIFE ASSUMED. BASIS FOR COST -
- INTEREST AT 4.25%COSTS ESCALATED TO 1970.

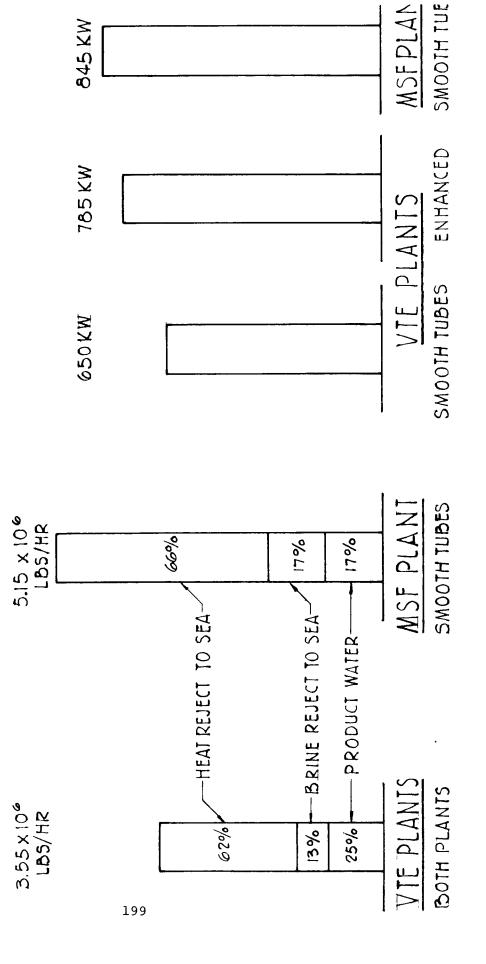
### CAPITAL COSTS SMALL (2.5 MGD) PLANTS



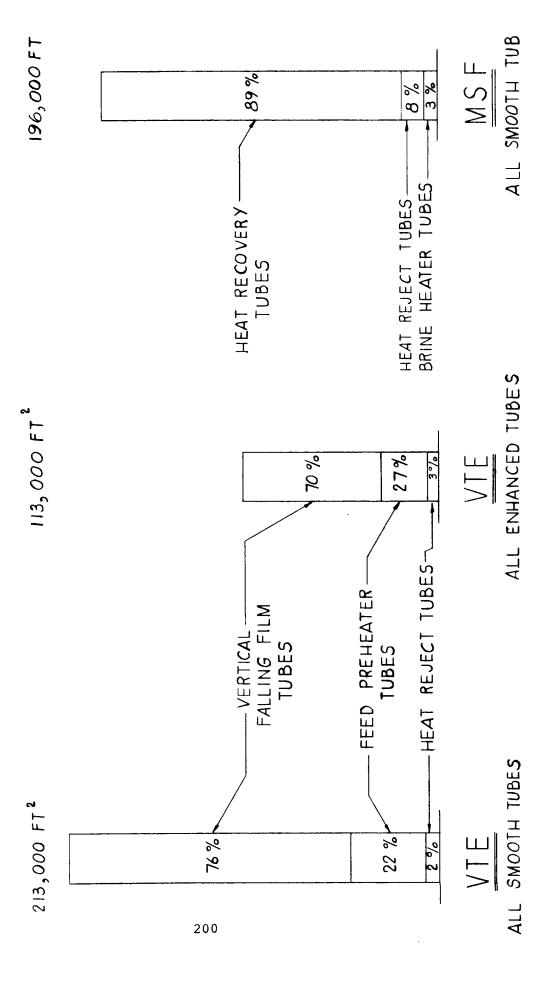
# SMALL (2.5 MGD) PLANTS

# SEAWATER INTAKE FLOWS

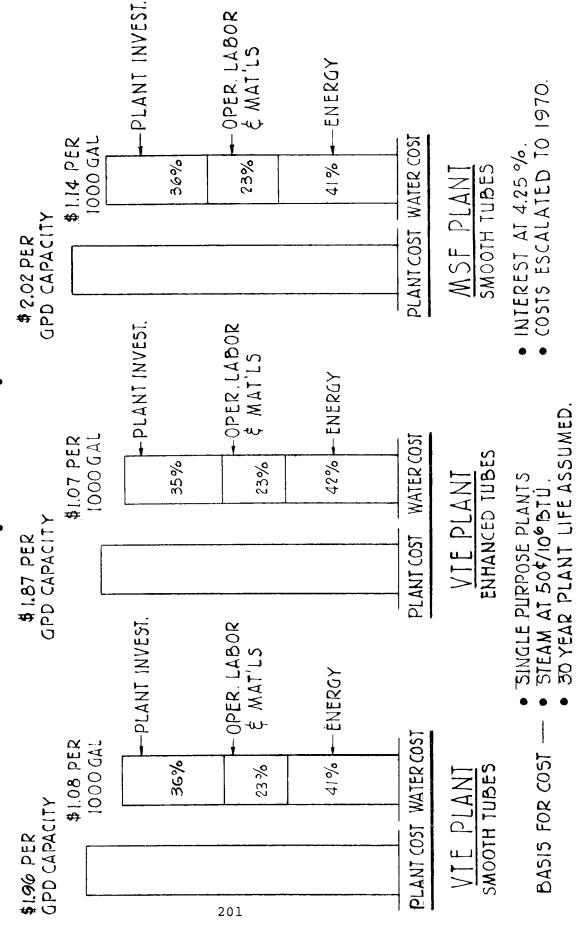
# PUMPING POWER REQUIREMENTS



### HEAT TRANSFER AREAS SMALL (2.5 MGD) PLANTS



## CAPITAL AND WATER COSTS SMALL (2.5 MGD) PLANTS SMALL



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SEA WATER DELIVERY	66.226	LNI	INTAKE PUMPS	6.5421
	162.236	PHT	PHTR FEED PUMPS	9.9828
PRODUCT DE	108.575	PRO	DUCT PUMPS	4.0098
BLOWDOWN DISCHARGE	40.000	8 B C D	MOOMN PUMPS	0.9841
DEAERATOR BOOSTER	12.247	8008	STER PUMPS	0.2851
		1943	EFFECT PUMPS	14.0817
		TOTAL	AL	35.8857
IV SUBSIDIARY SECTIONS				
	BRINE HEATER	CONDENSER	OEA	DEAERATOR
LMDT, F	13.045	18.412	TEMP IN-HOT, F	97.000
H INSIDE, B/HR FT2 F	2067.93	1021.95	TEMP IN-COLD, F	65.000
H OUTSIDE, B/HR FT2 F	2648.65	1714.87	_	77.000
DVERALL U. B/HR-ET2-F	554.160	394.748		66.000
HEAT RATE, STU/HR	0.2016140 10	0.4574090 10	IN-HOT, LB	0.5438080 08
FOULING RESISTANCE	05000-0	0.00010	FLOW IN-COLD, LB/HR	
SURFACE AREA, FT?	0.2362600 06	0.6430810 06	FLOW DUT, LB/HR	0.1437200
NO OF TUBES	58017.	27716.	VAPOR TO EJECT	0.1696000 05
TUBE LENGTH, FT	20.740	88.626	INERTS, LB/HR	0.1756250 05
TEMP-COOLANT EXIT, F	~	103.717		
TUBE FLOW	0.1437200 09	0.126734D 09		

19.2 - 250 EGD VIE COMPUTER PRINTOUT

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	σ	1428.7			187.19	138.73	194.72	0.47280 07	0.43480 06	3.43480 05		2.4008	360.3378
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	4	10701	0.41470 08	1 0.06731	٦.	76.44	139.03	0.45630 07	0.40180 06	0.40180 05	5 10.87	3.0838	417.7217
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_	18	#BRINE TE	OF WB	0EG F	HB	#BRINE	HB#BRINE ENTHALPY	OF WB, BTU/LB	_	#SATURATED VAPOR TEMP,	VAPOR TEMP	P, DEG F	
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VE & PHTR SHELL	)	0.936777620-02	0.109289130		
DNA	37042730	0.653933090-02			
	0.326060570 05	0.397029490-02			
VALVES AND PIPING	7470068D	0.334490830-02	0.390233600 07		
CHEMICAL CAPITAL	72300670	0.209802870-02			
INSTRUMENTS	0.214580620 05	0.261285290-02			
FLECTRICAL	.120243040	0.146414610-02	i		
DEAFRATOR	3726089D	0.117779070-02	0.137406900 07		
CONDENSER		0-156457150-02	0.182530680 07		
FR.	04870170	0.614757990-03	0.717207180 05	:	
SITE, BLOGS, CRANES	0.667276280 05	0.812512700-03	0.947917640 06	COST-SDIREIND	8/GPD
				- 1 1	
TOTAL CAPITAL	0.444766500 07	0.541572420-01	0.631824900 08	0.766838350 08	0.306735340
RETUBING	0.442614010 05	0.538951690-02			
HEAT	0.311527720 07	0.379333650-01			
CHEMICAL	0.961842470 05	0.117119345-01			
POWIR	0.565845110 06	0.689004805-02			
ວ	0.660053620 05	0.803718370-02			
MAINT & SUPPLIES		0.55024678E-02			
EJECTOR STEAM	0.665950050 05	0.810898190-03			
TOTAL OPERATING	0.582971610 07	0.709858970-01			
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COST FACTORS	Į.	 			
HEAT %\$/MMBTU< 0.05906	PLANT LIFE.	:	:		
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ANN CHARGE RATE 0.05800	PHTR TUBE L		PHTR	ETUBING CHGE RA	0
INTEREST RATE 0.04000 HIGHER COST FACTOR 1.21369	CNDNSR TUBE	LIFE, YRS 30.00000	OO CNDNSR	RETUBING CHGE	RATE 0.0
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4	m	0.12643600 09	-2884695D	246.60	51.4	48.	m	37.0	.730442D-	.828919	
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7	3	0.12042300 09	.2934719D 0	37.8	42.7	39.8	25.3	28-3	.8838750-0	. 863574	9
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0	0	0.0	.29347190 0	31.9	36.9	33.9	19.4	22.4	.100881D-0	.900115	0
10		0.11448920 09	.29763770 0	28.9	33.9	31.0	16.4	19.4	.1080910-0	.903023	0
11	0	0	.2976377D O	25.9	31.0	28.0	13.4	16.4	.1158430-0	.922477	0
12	. 0	0.0	29763770 0	22.9	28.0	25.0	10.4	13.4	.1242660-0	942424	0
13	· •••	0.10862950 09	.30320280 0	19.8	25.0	22.0	07.3	10.4	.133707D-0	944395	0
14	0	0	.3032028D 0	16.8	22.0	18.9	04.3	07.3	.143907D-0	.96552	0
15	0	0.0	.3032028D 0	13.7	18.9	15.9	01.2	04.3	.1550490-0	.98720	0
16	3	0-10284010 09	.31022300 0	10.6	15.9	12.8	98.1	01.2	1676610-0	98796	О
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20	0	0	.3189264D 0	97.9	03.4	00.2	5.4	88.6	.2325710-0	10592	į.
212	0	0.0	.3189264D 0	7.46	00.2	97.0	82.2	85.4	253486D-0	10854	
22		0.91456850 08	32974150 0	4.	97.0	93.7	78.8	82.2	-2777430-0	108136	
23	0		.32974150 0	8.0	93.7	4		78.8	3045580-0	110967	0 0
24	0	0.0	.32974150 0	94.7	90.4	87.1	72.2	75.5	.3345260-0	113885	
25	m	0.85855250 08	.3433645D O	1.2	87.1	83.7	68.7	72.2	*369834D-0	112976	
26	C		34336450 0	77.8	83.7	80.3	65.2	68-7	0-0694604	11614	1
27	0	0.0	.34336450 0	74.3	80.3	76.8	61.8	65.2	4540120-0	11942	
28	3	0.80307300 08	36089890	70.7	76.8	3,5	58.1	61.8	.5076880-0	11765	1
29	0	_	.36089890 0	67.0	73.2	69.6	54.5	58.1	.5687420-0	12124	
30	0	0.0	.3608989D O	63.4	69.6	66.0	50.8	54.5	.6387770-0	.12495	
31	m	0.74806510 08	.3841455D O	59.5	66.0	62.2	47.0	50.8	.7251120-0	.12165	
32	0	0.0	.38414550 0	155.6	62.2	58.3	43.1	47.0	.8255450-0	.12574	l
33	0	•	.38414550 0	151.7	58.3	54.5	39.2	43.1	.9430750-0	.12998	
34	•	0.6934364D 08	.31217780	148.	54.5	1.4	3	39	.1063000 0	1497	•
35	0	•	.31217780 0	145.2	51.4	48.3	32.9	36.0	.1190800 0	.15366	
36	0		.31217780 0	142.0	48.3	45.1	29.7	32.9	.1337450 0	.15768	
37	0	•	.31217780 0	138.8	45.1	42.0	26.6	29.7	.1506310 0	.16183	
38	4		.34739860 0	135.3	42.0	38.5	23.1	26.6	.172480D O	.15617	ĺ
39	0	•	.34739860 0	131.8	38.5	35.1	19.6	23.1	.1982780 0	.160862	
40	0	. •	.3473986D O	128.2	35.1	31.6	16.1	19.6	.2288570 0	.165690	
<b>+</b> 1	0	•	.3473986D 0	124.7	31.6	28.1	12.6	16.1	.265297D O	.170656	
42	<b>+</b>		.4039052D 0	120.	28.1	4.1	8.5	12.6	.3155490 0	.1563	
43	0	•	.4039052D O	116.5	24.1	20.0	04.4	08.5	.3792830 0	161831	
44	0		.4039052D O	112.3	20.0	16.0	00.3	4.40	.459305D O	167402	
45	0	•	.40390520	8.2	16.0	12.0	96.2	00.3	.5608330 0	172998	
46	2		.38360000	3.9	12.0	08.1	4.	96-2	.6942940 0	182758	•
47	0	•	.3836000D	9.8	08.1	0	8.5	4	.8551370 0	.187786	
48	0	0.0	.3836000D	95.84	•	00.4	84.72	88.58	0.1063360 01	0.189888	0 01
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AL PHA		.5074970
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AS	0.5523150 05 0.5542200 05 0.5542200 05 0.5542200 05 0.551950 05 0.5746370 05 0.5885180 05 0.6029250 05 0.6029220 05 0.762920 05 0.762980 05	35930
WO	10.98 10.91 10.93 10.92 10.92 10.92 10.92 10.92 10.92 10.93	10.
9	2773.72 2755.93 2757.82 2757.82 2757.82 2757.82 2757.93 2719.98 2713.61 2714.65 2714.65 2714.65	113.2
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NO O	0.40301188D 09 0.40301188D 09 0.40566724D 09 0.40566724D 09 0.41278899D 09 0.41278899D 09 0.41278899D 09 0.4128899D 09 0.42482830D 09 0.43266534D 09 0.53304876D 09 0.53865776D 09 0.55865776D 09	327576D 0
C2	0.03485 0.03486 0.03496 0.03540 0.03540 0.03540 0.03540 0.03562 0.033662 0.03382 0.03382 0.03382 0.03382 0.03382 0.03382 0.04421 0.04422 0.044	0840
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13	3 0 0 0 3 3 6 0 0 0 0 3 3 6 0 0 0 0 0 3 3 6 0 0 0 0	•0336
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M 2 0.584360210 08 0.584574450 08	TCF 0.336000000-01 0.207947630 03 0.221975920 03 0.231614190 10 0.143719690 09 0.669023620 10 0.467409430 10	70F 0.467409430 10 0.510275760 10 0.336164390 10 0.644825950 10 0.448669170 02
M 2 0.58436	TCF 0.3360000	TCF 0.46740943

0.10910269D 03 0.10910269D 03 0.10910269D 03 0.10910269D 03

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10 0.336164390	0.190000000 02	0.190000000 02	0.190000000 02	0.190000000 02	0.190000000 02	0.190000000 02	0.19000000000000
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TCF 0.467409430	0.165342210 03	0.165342210 03 0.655600110 05 0.190000000 02 0.14138644D 01	0.138983830 03	0.15250194D 03 0.60468700D 05 0.19000000D 02 0.14138644D 01	0.152501940 03	0.128190510 03	0.149011060 03

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00 00 00 00 00 00 00 00 00 00 00 00 00											0.467400430.10	04 001.01.01.0					0.703459410 02	0.230727090 09	
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.2 - 250 MgD VTF	2 0-141386440 01	141386440 01	141386440 01	14138644D 01	141386440 01	141386440 01	141386440 01	141386440 01	141386440 01	141385440 01	0.201614100 10		0.644825950 10	.14138644D 01	141386440 01	141386440 01	10 063888826.0	0.13371588D 03	8 0.852574880 08
G,	0.19000000 02 0	0.1900000000000000000000000000000000000	0.190000000 02 0	0.1900000000000000000000000000000000000	0.190000000 02 0	0.190000000 02 0	0.19000000D 02 0	0.190000000 02 0.	0.1900000000000	0.190000000 02 0	7947430 03 0.221975920 03 0.201414190 10 0.143719490 09 0.469023420 10 0.467409430 10		0275760 10 0.336164390 10 0.644825950 10	0.190000000 02 0.	920 05 0.190000000 02 0.141386440 01	0.190000000 02 0	0000000 03 0.801599820 01 0.328988890 01 0.126734270 09 0.467409430 10 0.703459410 02	0000000 02 0.336039650-01 0.103716880 03	739087D 07 0.57462205D 08 0.85257488D 08
	0.590845290 05	760 05	760 05	0.276353150 05 (	60 05	0.0	80 08	0.580163570 05	0.590163670 05	350 05	0.20		0.51	0.580166920 05	0.580165620 05	0.2771.54900 05	0.76	0.65	0.51
	0.125256140 03		0.145889450 03	0.12267770 03	63		0.12317 295 03	0.1463:7:60 03	0.1463;7160 03	0.122991690 03	TCF 0.236000000-01		TCF 0.46740943D 10		0.1463:7980 03	0.122992380 03	TCF 0.245731640 03	TCF 0.334648810 02	TCF 0.893558R3D 08
	ł			1							- [		1			- 1	- !		ļ

CARD 1 0.1003D 01	0							
-		TRAINS	AMODJ	z	EVAP STAGE	REJ STAGES NO. ITER	NO. ITER	FLAG
		0.25000 09 0.40000 01	01 0.40000 01 0.15000 02 0.45000 02	0.15000 02	0.45000 02	0.50450 01 0.0		0-1000D 01
	TBRINE	щ Н	188		10	DEL TC	2	
	0.25740 03 0.26000 03	03 0.11200 03	1	0.9282D 02 0.1703D 02 0.6500D 02	0.65000 02	0.50000 01 0.25000 01	0.25000 01	
EFF.	ELEC	HEAT	CHARGE	RATE INT WPL		유	HBL	A q
CARD 3 0.8900D	-00002-0 00 C	0.8900D 00 0.2000D-02 0.6000D-01 0.5800D-01 0.4000D-01 0.3000D 02 0.3500D 02 0.4000D 02 0.0	0.58000-01	0.40000-01	0.30000 02	0.35000 02	0.4000D 02	0.0
000	SURFF	VTUBLN	00	WALL E	200	WALLC	S	
CARD 4 0.30000 01 0.12730 01 0.1000D 02	1 0.12730 01		0.7500D 00 0.3500D-01 0.1000D 01 0.4900D-01 0.1900D 02	35000-01 0.	10000 01 0.	4900D-01 0.	19000 02	
	89	BIKT BK	BKTC TBL	IFV TBLI	TBLIFV TBLIFP TBLIFC AIV	C AIV	AIE	AIC
CARD 5 0.1250D 03	0 03 0.26000 02	0.26000	02 0.15000 02 0.30000 02 0.30000 02 0.17180 01 0.12060 01 0.15620 01	0.30000 02	0.30000 02	0.17180 01	0.12060 01	0.15620 01
00	СРР	CAP F	FOUL V	FOUL PH	FOUL C	;	1	:
CARD 6 0.33600-01	- 1	0.2500D-04 0.9000D 00	0.0 00	0.5000D-03	0.70000-03	0.5000D-03 0.7000D-03 0.0	0.0	

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SURMARY OF PLANT DATA			
GENERAL		PERFORMANCE RATIOS	
PLANT CAPACITY, MCD	~	LOGAL	. 849
SEA VALUE CONCENTRATION	0.03360	1	12.8631
PROPUCT CONCENTRATION	•	R RATIO P	
DEAERATED FEED CONC.	•	LEVAP TU	.432
CONCENTRATION RATIO	2,0000		,
NO OF STAGES, EVAP	87		
NO STAGES IN REJECT			
TEMPERATURES - DEG F		MT/44	•
CHEAN	256. 200	r .	.712355E 0
IN E WE WE WE WAY		TURE FINA EVAP	.624547E
TOUR TENED	103.47		270154E 0
	0000	TOTAL PREDUCT	.862575E 0
PRODUCT	90.03	MOTH BLOVE	452096E 0
FEF	621.16	FUECTOR SYSTEM	.620823E 0
	65,000	NACONO	.861933E 0
•		SEATHTAKE	0,304274E 09
BRINE INTO HEATER	238,761	SEAWATER RETURN	.125594E 0
II DESCRIPTION OF PHYSICAL PLANT			
PLANT DIMENSIONS, FT			
DVERALL LENGTH	574.975	DVERALL FEIGHT	18,400
NO OF LEVELS	~.	2	2.700
HIDIS INVIA	4 74 580		
NO OF TRAINS	0.4	REJ SHELL VOLUMES FT	3 0.252684E 07
TUBING DESCRIPTION			
	I	æ	REJECT
DUTSINE DIA, INCHES	0.750	0,750	0.750
WALL THICKNESS, INCHES	0.035		640.0
K,BTU/HR FT2 F	26,000	!	26.000
FOULING RESISTANCE	0.00050		0.00070
FLOUDING FACTOR	19,000	19,000	19.000
NO OF TUBES	249197。		101953.
TUBE LENGTH PT	24.140	480,032	70.R03
SURFACE AREA, FT2	0.118118576 07	54E C8	0.14173621E 07
TUBE VECOCITY		0	2.00

DELIVERY 188.561 106.758 15CHARGE 40.000		
188.561 106.758 5E 40.000	INTAKE PLMPS	8.8360
106.758	S AND	1,5932
9E	UMPS	4,0343
	PUMPS	1.5105
	UMPS	1,8628
		3,2212
TOTAL		71.0580
IV SURSIDIARY SECTIONS BRINE HEATER	DEAERATOR	
HDT. F	•	91,586
F. B/HR FT? F	•	65,000
B/HR FT2 F	DUT. F	91,123
-F12-F	TO EJECTA F	70,000
	FLOW IN-HOT, LB/HR 0.14456	61E 09
ISTANCE	FLOW IN-COLD, LB/HR 0.27911	15E 08
SURFACE AREA, FT2 0-118119E 07	FLOW DUT, 18/HR 0,173025E C	25E 09
		<b>67E 04</b>
	LB/HR	90E 05

## 19.5 - 250 MGD MSF COMPUTER PRINTOUT

UPDATE MSF 250 HDG CURRENT TECHNOLIGY 1-24-68 DEPTH#15 FOLL#.COOS

COST SUMMARY PAGE

PAGE

8 0.46989308E 0.9000 0.00000 \$/6PD • 2,0000 EVAP RETUBING CHGE RATE REJ RETUBING CHGE RATE BR HTR RETUBING CHGE RATE COST-\$DIR+IND 0.11747327E 09 S/SQ.FT. PHTR AREA LOAD COST-\$ DIRECT 0.18686778E 07 0.21110063E 07 0.14742054E 07 0.33411008E 07 0.92104672E 07 0.42658516E 07 0.49491125E 07 8 G.97412572E 0.49579033E 0.50869837E C.21088140E 0,17995611E 30.0000 30.00000 30.00000 2.0000 INTAKE COST 30,00000 0.158550056-01 0.375842316-02 0.435040926-02 0.185795756-02 0.18598946-02 0.18598946-02 0.682156786-02 0.89504275E-02 0.8582522E-02 0,85825236-01 0,10260011E 0C 8 0,436815436-01 0,448188055-03 0.64466483E-01 0.13779107E-01 COST-S/KGAL 0,18842532E EVAP TUBE LIFE, YRS REJT TUBE LIFE, YRS BR HTR TUBE LIFE, YRS CHANNEL USED IN SEA AREA 0 \$/50.FT. VE 0 0 0.64260339E 07 COST-5/YR 0.35873467E 0.13020926E 0.36663371E 0.35809860E 0.13521028E 0.13521028E 0.1366479E 0.24174910E 0.1316091E 0.56022124E 0.73505394E 0.15474429E 0.70483962E 0.36807443E PLANT 0 0.1000 0.00100 0.06000 0.04000 TOTAL\_(CAP+RETURE FOPER) 0.0286 INTEREST RATE HIGHER COST FACTOR MISG. TOTAL\_UNIT COSTS \$/CU.FT. SHELL COMPONENTS TOTAL OPERATING ANN CHARGE RATE SITE, PLOGS, CRANES HEAT (\$/MMBTU) VALVES AND PIPING TOTAL CAPITAL POMER (S/KWH) VE + PHTR SHELL PUMPS AND MOTORS SEA-WATER INTAKE MAINT + SUPPLIES CHEMICAL CAPITALINSTRUMENTS RETUBING DEAERAIOR BRINE HEATER FACTORS ELECTRICAL OPERATING CHEMICAL POVER EVΔP COST HEAT 213

UPDATE HSF 250 MDG CURRENT TECHNOLOGY 1-24-68 DEPTH#15 FOUL\*.COC5

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STAGE	AREA DM		DPTOM	HOLD	CPADD	**************************************	. 5U . 317577077	450506
-	4105396	9	15616837E-	332183864		D 16 4 6 7 7 7 6 4	BOOR SOLE OF	24200753E 0
2 1	41719313	0	1675349FE-	35232:446-	0-1767.41CBE.	0.00044000E	82297476F O	23896449E
· •	5755554	0	10840277	0-36676176	J-1-071/705.	54863837E	84570765E 0	23592131E 0
	43011100	0	-32466011	01107-1110	34744704F-C	55499244E 0	86905374E 0	23247803E 0
· •	4461344	) C	17318022F	43640137E-	350730746-0	.56123456E 0	89300250E 0	,22943457E 0
•	45135141	) C	1746152RE	46157°37E-0	338933195-0	56769131E D	91741327E 0	22679093E 0
<b>a</b> 0	45896467	0	176064918-	48799978E-0	32732454E-C	.57423237E 0	94246159E 0	.22374716E 0
•	44696869	0	17753010E	.516C5225E-	,31603878E-C	0 38087963E 0	968/0/4/E 0	0 31760/0774
្ព	47507210	0	.17901073E	546r3577E-	305143606-0	.58761585E O	0 363636646	21441471F O
=	42358020	O	18050758E	.573384405-	29467083E-C	0 36766456.	105394566 0	21157013E 0
75	49243496	0	162021/35	-012/103/E-	275026155-0	0 100/1100°	107993936	20852530E 0
64-	71172224 511172224	a c	100004401	0475000E	26584743E-C	615718CZE 0	11096984E 0	20543025E.0
· •	52406523	C	18499655E	72471619E-	.25761982E-0	.62657457E C	.11403168E n	.20243549E 0
9	53902405	0	.18405669E	.75935364E-	.25042372F-C	.63941649E 0	11717421E 0	.19939087E 0
17	,55443144	O	.18311936E	.79612732E-	.24407315E-C	.65254332E 0	120404051.	0 360046061.
80 (	0.570F9437E	05	0,18219335E-01	0,63557129E-01	Ov	n n	0.123/2403E 01	0
م ا	(27) 17 (07)	9	. 161207/1E	0-1107750/0	プログラングラング 100mm	693300596	13063931E 0	1572099760
9.5	, 60, 70, 100, 100, 100, 100, 100, 100, 100	<b>O</b> C	179460625	948427936-0	22461471E-C	70738841E 0	13424376E 0	.18416406E 0
22	64333400	9 0	17858024E	101P6005E 0	.22080573F-C	.72158362E 0	.13793745E 0	.18111786E.0
23	,643473E4	0	.17770597E	.10723877E 0	.21729937E-C	.73605705E 0	14173984E 0	173r7126E 0
54	69446930	0	.17686177E	.11296844E 0	.21404962E-C	.75055550E 0	14543203F n	0 115,202/14 0
25	7064250C	a	.17602376E	11912537E_0	.21102250F_C	0 74/05/07/45	D JOSEPHANE	0 36.607.71.1
5¢	77930078	0	.17522793E	.12570190E	20818257E-C	.7949464E	1579558E 0	16578080E 0
, a	0777777	0 0	17449041	14035734F O	20296733E-C	80960184E 0	.16225675E 0	16283194E 0
0 6 6	80426767	<b>5</b> C	172979485	14850F15E 0	20054680E-C	.82449889E	.16648815E n	.15978250E 0
'nΨ	83179160	0	17226885	.15722556E 0	.19823052F-C	.83958549E 0	.17125843E 0	.15673249E 0
<u>.</u>	86029653	a	.17154556E	.16658020E_0	.19600926F-C	85519615E O	17599635E n	15368183E 0
32	94980046	0	.17100927E	.17687225E 0	19384883E-C	.86885336E 0	.IROSBII7E O	.15063021E 0
6	92072004	0	170487465	.18751199E 0	189649405-0	.87202460E 0 .89509296E 0	10035433E 0	14457444E 0
) (r	0921256	<b>5</b> C	169715426	21256697E 0	18762777E-C	.90763986E 0	.19534919E 0	.1414700AE 0
36	10209353	0	.16950927E	.22775677E 0	.18559829E-C	.91822232E 0	200319336 0	.13841438E 0
37	10572434	9	,16932251E	.24275970E 0	18358069E-C	.92881430E D	20543765E 0	13535748E 0
w (	10944545	0	.169371906	.25988770E 0	.18155783F-C	0 36700056 0 0 367361349	.21042103 .21561136	.1362384E 0
7 4	24546611	0 0	9/2644601•	0 366476666	7-300026671	94753146E O	22054026F 0	12617659E 0
<b>,</b>	12147232	<b>)</b> C	17027067	322C9015E 0	175751756-0	95080453E 0	.22560140E 0	12311237E 0
45	12563302	9 0	17116724E	34724698E 0	.17408213E-C	.94797010E 0	.23022112E 0	.12004511E 0
4	12999764	0	.17210011E	.37607574E 0	1725373nF-C	.94494197E 0	23493901E 0	.11697538E 0
75	13426170	0	.17371864E	.40861511E 0	,1710732	.93315690	.238F588RE 0	.11350130E O
	13872997	0	175395876	.44456482E O	1690601	896627745E O	0 3/646/342	0 30862-011.
0 1	10616241	0 6	-1/011207E	5048464F	16691001	.89353246E O	24761504E 0	10467832E 0
• 4	1630661	. C	157601115	50250244E 0	16554981	90067936E 0	24412372E 0	10164107E 0
4	00086666	) C	13912240E	49745015E 0	16240249	.84138751E n	43684147E 0	.95841657E 0
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DEPTH#15

1-24-68

TECHNOLOGY

CURRENT

HOC

250

ASF

UPDATE

Page 7

UPDATE MSF 250 MDG CURRENT TECHNOLCGY 1-24-68 DEPTH#15 FOULW.0005

HE CALCULATION OF NO DEAERATORS HAS BEEN NFTICINED FOR THIS CASE  JNDRFLOW AT 71440 IN MQ  JNDRFLOW OF CO 0.14138646E 01  JNDRFLOW AT 71440 IN MQ  JNDRFLOW AT 714138646E 01  JNDRFLOW AT 71	RSENCE BASED ON AMP DATA AS USED IN DRVF CODE
THE CALCULATION OF NO DEAERATORS HIS BEEN NITIONED FOR THIS CASE  NUMBELOW AT 71440 IN MO  NUMBEROW AT 71440 IN MO  NUMEROW AT 71440 IN MO  NUMBEROW AT 71440 IN MO  NUMEROW AT	
NORFICH AT 62360 IN HQ   NORFICH AT 71440 IN	CALCULATION OF NO DEAERATORS HAS BEEN OFTIONED FOR THIS CAS
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NUMBELDW AT 72440 IN MO 0.6254132E 03 0.24007370E 06 0.190000000E 02 0.14138446E 01 0.6254132E 03 0.2009434E 06 0.19000000E 02 0.14138446E 01 0.6254132E 03 0.2009434E 06 0.19000000E 02 0.1413844E 01 0.6253132E 03 0.2479435E 06 0.19000000E 02 0.1413844E 01 0.6253132E 03 0.2479707E 06 0.19000000E 02 0.1413844E 01 0.6253672E 03 0.2479707E 06 0.19000000E 02 0.1413844E 01 0.6253672E 03 0.2479707E 06 0.19000000E 02 0.1413844E 01 0.6253672E 03 0.24919737E 06 0.19000000E 02 0.1413844E 01 0.6253672E 03 0.24919737E 06 0.19000000E 02 0.1413844E 01 0.6253672E 03 0.24919737E 06 0.19000000E 02 0.1413844E 01 0.6253677E 07 0.16410040E 03 UNIT WICTH # 0.11864510E 03 0.14074627E 07 0.65772848E 07 0.65742613E 06 0.23217645E 01 2 0.1974837E 07 0.65772848E 07 0.65742613E 06 0.23217645E 01 2 0.11974837E 07 0.26671355E 07 0.1799584E 07 0.89412020E 02 0.30340000E 02 0.11284739E 2 0.11974837E 07 0.26671355E 07 0.1799587E 03 0.23799999E 02 0.30340000E 02 0.66896172E 02 0.60331012E 02 0.30340000E 01 0.66758242E 301 0.18799999E 02 0.25000000E 02 0.66896172E 02 0.60331012E 02 0.30340000E 01 0.6626293E 302 0.16243614E 02 0.51593220E 02 0.71657999E 02 0.303427415E 09 0.	FLOW AT 71440 IN
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TEN CHANNEL USED ANNUAL COSTS AKE 0.38720034E 00 VS 0.3908324]E	DPEN CHANNEL USED ANNUAL COSTS ARE 0.38726034E 06 VS 0.39683241E 06
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UPDATE MYF 250 MDG CURRENT TECHNOLOGY 1-24-68 DEPTH#15 FOUL#.0005

ı	CARD 1		CARD 2		CARD 3		CARD 4		CARD 5		CARD 6		CARD 7		CARD 8		CARD 9
CASE NO.	0.3000E_01	7.5	0.2542E 03	MUDU	0.1000E 02	EFF"	0. BAOOE 00	HUQ	0.7500E 00	BIKTH	0.2400E 02	T of	0.5000E-03	CH	0.3500E 02	CPSTB	0.1500E 02
ی	0.2500E 09	THF	0.2500E 03	ALEV	0.2000E 01	ELEC	0.1000E-02	WACLH	0.3500E-01	BIKTE	0.2600E 02	RE	0.5000E-03	풒	0.2500E 02	FACID	0.2674E 01
æ o	0.1285E 02		0.918CE 02	TINO	0.400CE 01	HEAT	0.100CE 00	006	0.750cE CC	BIKTR	0.260CE C7	er ex	0.7000E-03	HBL	0.400CE 02	ACACID	-0.1222E-01
a	-0-		0.6500E 02	TOTALE	0.4800E 02	CHARGE	-0.6000E-01	WALL E	0.3500E-01	CSLTBH	0.3000E 02	SH	0.1900E 02	HPR	0.9200E 02	DPLIM	0.1470E 02 -0.
σ	-0-	41	-0-	TOTAL R	0.200nE 61	RATE INT	0.4000E-C1	DOR	0.750nE 60	CSLTBP	0.300nE C2	SE	0.1900E CZ	FFHL	-0-		
a a	C.2000E 01 -0.	TA	C.8000E 02	FREEBD	C. 2700E 01	764	C. 3000E 02	WALLR	C.4900E-01	CSLTBR	C.3000E 02	SR	C. 1900E 02	SSF1	•0•		-0-
RDE	-0-	8	0.3360E-01	TRAYDP	0.400E 01	CAPF	0.9000E 00	A3	0.2000E 01	SFI	0.1000E 01	Y.	0.4500E 01	SSF	0.1000E 01	vs.co	0.1300E 01
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STAGE FT/SFC			•		11				-
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60.00	C.99337F 0	31,277	.725586	.2551RE 0	2.13763E	#05/0E	0.47785E 0	2,695	aō c
4 10.549475	0.990083E 08	31.32/042	1.742230 0	24142E 02	0 0 13154E 0	0.79350F 0	0 0.44436F 0	2.708	~ 60° C
11.0133	0.9814766	31.427	777130	215705	10000	787516	0 10110100	2 2 736	
11.5009	C.976913E 0	31,4778	7:5473	20370E 0	0.11445E	78159E	0.10540E 0	2 2.751	٠.
12.0135	0.972375E 0	31,528	.814427	.19225E 0	0.1C915E 0	0.77574E	0.10753E 0	2 2.767	٠.
12.55264	0.967864E 0	31,579	1 060469	.18134E 0	0.104C2E 0	0.76996E	0.10973E 0	2 2,784	ç
1197	0.963377E 0	31,630	. 8£4299 (	.17094E 0	0.95081E 0	0.76425E	0.11202E 0	208.2	•
13,7162	C.954914E_0	31,681	. 675300 C	161035_0	0.9943156_0	0.75860E	0.11439E 0	2 2.520	9
14.34394	C.954474E 0	31,733	086799	.15160E 0	0.89729E C	0.75301E	0.11685E 0	2 2.640	7
0.00.01	C.950C5FE 0	31.785	919623	.14767E O	P. RSSCAE	0.74750E	0.11940E 0	2 . 2 . 861	∵:
10.0.01	0.3600360	01.000	7 6666644	. 13409E O	0.0103/E 0	0.74204E	0.127046	7000	
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18.01438	C.932407E O	31.995	7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	0 100011	0.44219E	726055	0.12/005	7.956	27
18.87104	C.928296E 0	32.0491	045186	10407E 0	0.656208	0.720836	C. 13369E 0	2 2.982	
19.77286	G.924003E O	32,1030	073711	0 1.0.2.0.0.0	0.621416	0.715685	0 13689F 0	3,010	
20.72387	G.919729E 0	32,157	102265	91292E 0	0.58816F	0.71059F	0.14021E 0	3,039	• "
21.72507	0,915473E 0	32,2116	132402	85410E 0	C. 55624E	0.70555E	0.14366E 0	3,069	4
22.7 <sup>4</sup> 110	C.911233E 0	32,2669	164094 0	.79A4BE 0	0.52576E 0	0.70057	0.14725E 0	2 3,101	٠.
23,89330	_0.907010E_0	32,322	126994	0 30657L	0 3754576	0.69565E	0.15098E_0	3,135	S,
22.06656	0.9025038	2,378	.231221	.69425E 0	0.46870E 0	0.69078E	0.15486E 0	3,170	1.57
17:5:07	0.0000000000000000000000000000000000000	[ C 6 4 6 7 ]	1 050/42	.04934E 0	0.442F3E 0	0.683966	0.15491E 0	2 3.207	•
10000	0 0154460	77/4	00(+)6	0 3/1/204	O. 41000E O	0.081ZnE	0.16311E 0	2,245	0,
20.424	0.8861136	,		0.0001E 0	0.35243E 0	0.01049	0 100401 0	707.6	- 1
31,95521	0.881971E	6671	67777	48733F	0.34725F	0.667275	0.17680F 0	37.5	•
33.54690	0.877845E	2. 726R	471795	45260E_0	0.32655E	0.66766E	0.18172E G	3.421	
35.29596	0.87371PE 0	17871	.517366	410946 0	0.3c66/E 0	0.65815E	0.18589E 0	3.472	٥.
37.10970	0.569406E 0	2.8481	564393	.3892AE 0	0.28770E 0	0.653695	0.19229E 0	2 3.524	٥.
33.99670	C,865503E 0	5606.	.616409	.36051E 0	0.26974E 0	0.6492RE	0.19792E 0	2 · 3.578	9
40.98909	C.861407E 0	.9726	.670924	33353E 0	0.25275E 0	0.64492E	0.20378E 0	2 3,635	~;
43.04878	0 857315E 0	3,0361	728372	3CA26E 0	D. 23660E C	0.64060E	3.20991E n	3.694	~
10000 C4	0.85322EE 0	1000	758671	.28460E 0	0.22131E 0	366969.0	0.21532E 0	3,755	٣,
50.11.05	0 10404040	1001.	101760	0 000000	0 11 11 10 10 10 10	0.05711E	0.263635.0	0000	•
52.7333	0.8409890	7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	700016	0 241142	0 18005F 0	0.02/735	0.237645	7000	•
55.40305	0.8369000	26046	F-04-04-0	0 0047706	0 3/3/10/10	7.0630.0	0.23/405.0	0.63	ů٩
56.43259	0.632817F 0	4 4 4 4 4	10000	18771	0 2/0/01	2445	0 367646 0	200.4	• •
61.53276	0.628731E 0	5106	27175	172115	144786	0.61165	0.262225	20107	
64.84635	0.82464CE 0	1,5834	305252	15764E	13435F 0	0.50769F	0.27133E 0	4.254	- ~
68,34150	0.820545E 0	3,6575	306305	14420E	12412E 0	0.60377E	0.28138E 0	4.333	•
.22946	0.816444E 0	1.7332	482514	13179E	11474E 0	0.59989E	0.29171E 0	4.408	3.05
76.28590	0.812339E 0	1,8103	577277	,12031E	1c530E 0	0.59605E	0.30354E 0	4.486	7
RO.87438	0.808226E 0	8888	644875	1097RE 0	0.96018E-0	0.59225E	0.31486E a	4,552	~
R6.275#2	0.804110E 0	0.96.1	669222	1001RE 0	0.85526E-0	0.58849E	0.33466E 0	4.594	~
73.74113	0.8000001	1000	566751	91630E	0.97032E	0.58477E	0.31320E 0	4.545	ď
C+0.5.70	0 1010/6/6	1000	9051257	0 3/7/10	0 146434	100 BC 0		-	•

(.) - 250 MGD MSF COMPUTER PRINTOUT

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UPDATE MSF 250 MDG CUPRENT TECHNOLOGY 1-24-68 DEPTH#15 FOUL#,6005

TUBE RUNDLE IN EVAPORATOR SECTION

200 20 0.130000E 0,130000E m • . S/D RATIO RUNDLE WIDTH + S/D RATIO BUNDLE WIDTH 0.72174072E-02 0.78849792E-02 0.85525513E-02 0.44837952E-01 0.49167633E-01 0.53497314E-01 0.53873291E-01 0.22521973E-01 0.22521973E-01 0.24612427E-01 0.26702881E-01 0.34667969E-01 0.34667969E-01 0.37658691E-01 0.12973785E 00 0.76316833E-01 0,16750336E 00 0.914001466-0 0.16151428E-C 0.41248322E-0 0.19081116E A.18115997E 0.27049255E 0.14953613 0.13755798 0,17616272 0.29190063 CTCPO 200 200 0.11734009E-01 0.12798309E-01 0.13862610F-01 0.19546509E-01 0.21411896E-01 0.23277283E-01 0.419859E 0 0.40000E 0 0.255452E 0.66413879F-02 0.71716309E-02 0.25520325F-01 0.27763367F-01 0.30475616F-01 0.47554016E-01 0.397186286-01 0,52307129F-01 0,57060242E-01 0.1644R975F-0 0.179977426-0 0.331878666-0 0.68199158F-01 0.88964370E-01 0.10530472F 0C 0.282500E 0.62429700E-0 0.364532476-0 0.436322F-0 0.78086853F 0.13296509E 0.170516976 0.14268112F NC OF TUBES/BIDLE
HCRIZ ROWS
BUNDLE WIDTH, FT
DPCLR NO OF TUBES/BYDLE HERIZ ROWS BUNDLE WIDTH, FT TUBES/BNDLE RIWS 0.4497848E-02 0.45928955E-02 0.47429459E-02 0.5047446-02 0.50474465E-02 0.29920936E-02 0.30905008E-02 0.31974316E-02 0.33043623E-02 0.22063255E-02 E-02 0.41705966E-02 0.43766740E-02 0.25006970E-02 0.27074814E-02 40345192F-02 0.665603586-02 0.53502457F-02 0.55165556F-02 0.61599612F-02 0.6448c6n3F-02 0.65732261E-02 0.66995919E-02 0.39n52963F-P 0.632401676-02 0.35330057E-C .36545396E-0. 0.67437693F 0.3776735 0,564535928 0,58521628E 0.579016115-02 0.734710595-02 0.F6593628E-02 0.121307375-01 0.287628176-01 0.111923225-01 000 0.24154663F-01 0.264597406-0 0.999755866-01 000 0.500323495-0 Q. 13256073E-0 0.143814095-0 0.15727997E-0: 0.18692017E-0 0.203094485-0 0,22232056F-0 0.31551361E-0 0.34339905E-0 .40817261E-0 0.52825928F-0 0.57220459E-0 .62198639E-0 2647095E-0 0.750000E 0.690000E 0.484715E 0.750000E 0.730000E 0.512960E DTIMIT SECTION 0.29944646 E-02 ( TUBE BUNDLE IN REJECT 0.18301872E-02 0.25034516E-02 0.14123838E-02 0.151428465-02 194911095-02 0.256229A9E-02 0.391448F1E-02 0.376933846-02 TURE DA, INCHES 0.16130313E-0 0.20124576E-0 0.32554152E-0; INCHES 0.15434589E-0 0.16454398E-0 0.17182484E-0 0.221109566-0 0.22823376E-C 0-351756567 0-350162710 0.28248758E-0 0.29094676E-0 0.33432517E-0 0,343579406-0 0.35273344E-0 0.39101409E-0 0.40213BRBE-0; RUNDLE HT, FT BUNDLE HT. FT 0.177421535 214344521 0.39748182 0,36102656 0.36931971 TURE DO, I VERT ROSS DPIVIT

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19.4 - 2.5 MGD MSF COMPUTER PRINTOUT THE FIUOR CORPORATION, LTD.

PAGE NO. 1					PCT		
PAGE	89-				3.500 WT. PCT	2898 MLB/HR	RECYCLE BRINE CONC. FWT. PCT 7.000
4334	118-68	S A A	45	E 250•	m	2898	RECYCL CONC
CONTRACT 4334	DATE	BY	TOTAL NUMBER OF STAGES	MAXIMUM BRINE TEMPERATURE 250.0 F	SEA WATER CONCENTRATION	WATER	INITIAL STAGE TTD,F 6.623
			TOTAL NUMBI	MAXIMUM BR	SEA WATER	EXCESS SEL WATER	BLOWDOWN TEMP*F
				16PD			BLOWDOWN MLB/HR 865
SALTNE HZO				. 2.49 MMGPD			RECYCLF RRINE•MLD/HR 4185
OFFICE OF SALT	WASHINGTON DC		.s. 1	865 MLB/HD.	65.0 F	1730 МСВ/Нр	PRODUCTION MLB/HP R 865
CUSTOMER OF	LOCATTON WA	RUM NO. 2	TOTAL NUMBER OF EFFECTS	OUUCTION	SEA WATER TEMPERATURE	R MAKEUP	EFFECT NUMBER NO. OF STAGES 1 42
			TOTAL MU	TOTAL PRODUCTION	SEA WATE	SEA WATER MAKEUP	₩ <i>2</i> ₩ ₩

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RUN NO. 2

## BRINE HEATER

Z57.00 (TER.F 13.38		75.58 75.58	5362	6.08	EAM 10.69	1765	7	15.47	.7500	0350	26	/SEC 6.000		
STEAM TEMPERATURE. F TEMPERATURE DISE OF BRINE THROUGH HEATER.F	LOG MEAN TEMPERATURE DIFFERENCE	HEAT REQUIREMENT, MMBTU/HR	AREA REGUIREMENT, SO. FT.	STEAM REGUIREMENT. MLD / HR	STEAM ECONUMY RATION BS PRODUCTZLB STEAM	TOTAL NUMBER OF TUBES	NUMBER OF TURE HUNDLES	LENGTH OF TURES.FT	OUTSIDE DIAMPTER OF TUBES, IN	WALL THICKNESS OF TURES, IN	THERMAL CONDICTIVITY BTU/HR-FT2-F/FT	VELOCITY OF DECYCLE PRINE IN TUBES, FIZSEC	PRESSURE DROP THROUGH THIS SECTION, PSI	

## PUMPS

78.0 93.0	3	149.2	183,4	58.5	449.0	878.3
	TOTAL DIFFERENTIAL	22.05	72.31 30.20	45°20	75.18	TOTAL
PERCENT • PERCENT	MISC. PPESSURE	16.00	16.00	16.00	16.00	
PUMP EFFICIENCY.PERCENT MOTOR EFFICIFNCY.PERCENT		SEA WATER PUMP	MAKEUP PUMP ALOWDOWN PUMP	PRODUCT PUMP RECYCLE PUMP	EFFECT NO. 1	

## EJECTOR STEAM

EJECTOR STEAM MLBZHR

	19.4 - 2.5	19.4 - 2.5 MGD MSF COMPUTER PRINTOUT		
1.7			HUN NG. 2	PAGE NO.
	HEAT R	HEAT RECOVERY	HEAT REJECTION	CTION
	RECYC! E	MAKEUP	HECYCLE	MAKEUP
MUNITER OF TUMES PER STAGE	2052	<b>0</b>	0000nc+	1331
MU WER OF THE MINNERS HER STAGE	-	0	0	
FEGTH OF TUNES. FT	11,16	11.16	21.00	21.00
OUISTDE PLANETER OF TURES+ 14	.750	.750	. 750	.750
SALL THICKNESS OF TURES. IS	0.35	• 035	• 035	• 035
THEREAL CONDICTIVITY RICKHELFIZER	26.0	26.0	24.0	26.0
VELOCITY IN TUNES. FIVE	000,5	000.0	000.0	000-9
IN-TIUBE PRESSURE DECP. PSI	27,11	00.0	<b>50.</b> C	0.44
TAC TORNER AND FIXTH LUCKER, DAT	1 17	00.0	00.0	.13