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Segregation in binary and polydisperse stirred media mills and its role on grinding effectiveness

Journal Pre-proof

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Abstract

Most industrial vertical stirred mills contain a non-uniform set of grinding media sizes. However, this fact is often ignored in simulations, which mostly use monodispersed media. The paper explores the **fundamental dynamics of vertical mills when using multiple sizes of grinding media**, by using DEM simulation. Our results suggest that by including both large and small media, one may be able to optimise its performance in several manners. The energy going into the contacts can be increased by including a second size, leading to more effective grinding. Including smaller media can also reduce the power draw of the mill, increasing the efficiency and sustainability of the mill. Finally, the natural segregation between different sizes creates different types of collision which grind different particle sizes more effectively. Segregation leads to smaller media at the bottom, so continuous processes can be optimised for fine grinding by feeding from the top, where the larger media are. This further enables control of the resulting product specifications.

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1 1. Introduction

Within industrial stirred mills, almost no system has a full set of identi-2 cally sized grinding media [1]. There are several reasons for this. The most 3 common is due to manufacturing, but the differences are also caused by age-4 ing and wear, particularly in continuous mills which are becoming 5 increasingly common in almost every industry [1-5]. de Bakker 6 [1], even stated that one of the most common errors in research 7 is using monodisperse media, which will give a higher specific en-8 ergy than reality. Because of the nature of the technique, media become 9 smaller over time until they pass out of the mill naturally. The lost ma-10 terial is replenished periodically to maintain overall media mass. 11 Altun et al. [6] investigated the effects of media wear on an in-12 dustrial and laboratory scale mill, observing noticeable differences 13 in the amount of wear caused by using different materials. Mill 14 operators may also deliberately choose to use multiple media sizes because 15 they have different benefits on the grinding performance [4, 7, 8]. Smaller 16 media produce a finer grind as stated by Napier-Munn et al. [9], and later 17 Edwards [10] because their increased collective surface area benefits finer 18 grinding. However, Jankovic [11] did state that there may be a deteriora-19 tion in mill efficiency below a certain size limit, presumably because there 20 becomes insufficient energy within smaller media. [9]. 21

Previous studies compare the milling performance of different media sizes 22 [12]. Mankosa, Adel and Yoon [13] were one of the first groups to do this by 23 investigating the effects of media size within a low-speed stirred mill (200-24 330 rpm). They concluded that the optimal ball size to particle feed size ratio 25 was 20:1 and although this was for dry coal, their conclusions still hold for 26 most conditions including slurries. Blecher et al. [14] subsequently demon-27 strated that smaller grinding media were more likely to follow streamlines in 28 the mill and, therefore, were more likely to influence grinding performance. 29 Meanwhile Kwade, Blecher and Schwedes [15] suggested that using different 30 media sizes within the same mill would be beneficial as larger media sizes 31 produced a finer grind at lower specific energies and smaller media sizes were 32 more effective at higher energies. Bel Fadhel and Frances [16] later agreed 33 with these conclusions [10]. 34

A recent review article by Kumar, Sahu and Tripathy [17] high-35 lights this by mentioning that media size distribution has an effect 36 on grinding but not citing any studies which investigate the effects 37 of having multiple size fractions present. Sinnott, Cleary and Mor-38 rison [18] did investigate the effects of a polydisperse size range but 39 only for a single media charge as part of a wider shape study. Jaya-40 sundara, Yang and Yu [19] investigated the effects of using different media 41 sizes in a horizontal mill computationally, but each charge was a single size. 42 This is just one example of a series of studies that have observed the effects of 43 different-sized media within horizontal setups [20–24]. Cho et al. [8] took this 44 one step further by comparing a monodispersed experimental mill with simulated binary and three-size bead distribution ratios. They concluded that a 46

binary system of the largest and smallest size was optimal, rather than includ-47 ing an additional intermediate size. More recently, Soni and Mishra [25], and 48 then Yari et al. [26] have simulated at least binary dispersity in a tumbling 49 mill particle distribution. While the former based their study around mill 50 speed and concluded that there was an optimum for reducing segregation, 51 the latter focused on the size ratio, finding that different regimes occurred at 52 different ratios. While there is a lack of findings which highlight the 53 effects of having multiple particle sizes, literature discusses segregation phenomena and theory in other processes [27–33], so the 55 effects can be compared to those found within the vertical mill. 56

While there has been significant research into the effect of particle size in monodisperse systems [12–15], there has, to date, been no detailed, systematic study of the behaviour of Vertical Stirred Mills containing binary or polydisperse media charges. This study aims to address this gap by investigating:

- The effects of having two sizes of media within a Vertical Stirred Mill.
- How changing the size ratio of two fractions affects grinding
 performance.
- How changing proportions of large and small media affects
 performance.

Simplifying the case to binary size distributions initially allows for deeper
 exploration and more definitive identification of the effects caused by different
 ratios compared to using a full polydispersed distribution. While grinding

would affect shape and surface properties which are highly important in industry too, this approach isolates the effect of size specifically and focuses on key findings. A few polydispersed simulations were
run towards the end of the study to see if observations were consistent across
more complex media size distributions.

76 2. Methodology

77 2.1. DEM contact model

DEM is based on Cundall and Strack's original description of the interaction between two particles in contact [34]. Two equations, defining the translational (Equation 1) and rotational (Equation 2) velocities, can be derived from the resultant forces and torques shown in Figure 1 [34–36].

$$m_i \cdot \frac{dv_i}{dt} = \sum (F_{ij}^n + F_{ij}^t) + m_i \cdot g, \qquad (1)$$

$$I_i \cdot \frac{d\omega_i}{dt} = \sum (R_i \cdot F_{ij}^t - \tau_{ij}^t), \qquad (2)$$

where F_{ij}^n and F_{ij}^t are the normal and tangential components of force, and m_i , v_i , I_i , ω_i and τ_{ij}^t represent the particle mass, translational velocity, moment of inertia, rotational velocity and rolling friction. R_i is the distance between the particle centre and the tangential contact point and g is the gravitational force [35–38].

LIGGGHTS[®] (LIGGGHTS v3.8.0, DCS Computing GmbH, Austria) [39], an open-source DEM package, was used to carry out the simulations. The model used was the Hertz-Mindlin no-slip method, and this defines the

normal and tangential forces of a single pair of particles in contact (Equations
3 and 4 respectively) [40].

$$F_{ij}^n = K^n \cdot \delta_{ij}^n - \gamma^n \cdot v_{ij}^n, \tag{3}$$

$$F_{ij}^t = K^t \cdot \delta_{ij}^t - \gamma^t \cdot v_{ij}^t.$$
(4)

Each equation is the sum of an elastic and dissipative component. K^n and δ_{ij}^n are the elastic constant and normal particle overlap and $\gamma^n \cdot v_{ij}^n$ is the normal dissipative force. The equivalent tangential terms, K^t , δ_{ij}^t and $\gamma^t v_{ij}^t$ are used in Equation 4. K^n and K^t can be calculated from input data and $\gamma^t v_{ij}^n$ [38]. A full derivation can be found in Appendix A [40].

97 2.2. Model setup

A template model was created and validated based on the pre-98 vious experimental results of Rydin et al. [41] and simulation work 99 of Daraio et al. [36]. In the spirit of open and transparent science, 100 the base model used for this work has been made freely available at 101 https://github.com/darhyme147/ligggghts_stirred_mill_template 102 such that other researchers may independently reproduce and ver-103 ify our results, and more generally benefit from the use of this 104 calibrated model in their work. For direct comparison, and valida-105 tion, with the experimental data reported by Rydin et al [41], their 106 geometry was reproduced and the simulation focussed on 10 mm 107 media at attritor speeds of 50 and 250 rpm (0.43 and $2.17 \,\mathrm{m\,s^{-1}}$ tip 108

speed). While the tip speed could be considered low for some ap-109 plications, in the context of food, this is a reasonable range that 110 agrees with previous studies [2, 42]. The geometry, shown in Figure 111 2, and the remaining parameters reproduce the long arms design 112 from Daraio et al. and are shown in Table 2. The mill contained 113 14.80 kg of media (55 % fill level) [36] which were modelled with the 114 properties of stainless steel. A restitution coefficient of 0.7 and 115 sliding friction of 0.25 were also selected from the previous work 116 [36, 43], while the rolling friction was set to zero; a reasonable 117 assumption for spherical media particles considered here [44, 45]. 118 The validation results are shown in Section 4. 119

Five simulation batches were run, each defined by the ratio of the two media diameters. The larger size was fixed at 10 mm, while the smaller varied between 2.5 mm and 7.5 mm. A timestep of 10^{-6} s was selected as this is 140 times smaller than the critical timestep obtained for the 2.5 mm, using the Equation proposed by Thornton and Randall (Equation 5) [36, 46],

$$t_{cr} = \frac{\pi \cdot R \cdot \sqrt{\rho/G}}{0.01631 \cdot \nu + 0.8766}.$$
 (5)

 ρ is the particle density and G is the shear modulus. The numerical proportion of the two media size fractions was also changed while maintaining the overall mass. With five size ratios, eleven proportions and five attritor speeds, a total of 275 simulations were performed and analysed. Each was run on a high-performance cluster using 4 cores of an Intel[®] Xeon[®] Platinum 8360Y CPU

132 processor [47].

133 2.3. Post-processing

134 2.3.1. Media segregation

Because a binary-size distribution was used, media segregation will be 135 present. Segregation is a phenomenon whereby particles of different prop-136 erties, in this case, size, fully or partially separate into individual fractions 137 [32]. Any physical property difference can cause segregation, including size 138 [25, 26, 48], density [28, 31], geometry [49], and surface properties [27]. Mix-139 ing indexes are one way of quantifying segregation and the Lacey mixing 140 index has been used in this study (Equation 6). To calculate this, the milling 141 region is divided into equal volume cells [50, 51], 142

$$M = \frac{\sigma_0^2 - \sigma^2}{\sigma_0^2 - \sigma_r^2}.$$
 (6)

 σ^2 is the variance of the concentration of a reference component within each cell of the system in question and can be expressed using Equation 7,

$$\sigma^2 = \sum_{i=1}^{N} \frac{(\phi_i - \phi_m)^2}{N - 1},\tag{7}$$

where N represents the number of cells occupied by media, ϕ_i is the local concentration of the reference component in cell i and ϕ_m is the global concentration of the reference component. For a binary system, like the ones observed here, the maximum variance, σ_0 , can be determined as

$$\sigma_0^2 = \phi_m (1 - \phi_m). \tag{8}$$

¹⁴⁹ Finally, the theoretical minimum variance of a cell, σ_r is defined by

with *n* being the maximum number of particles that can fit within an individual cell.

 $\sigma_r^2 = \frac{\phi_m (1 - \phi_m)}{n},$

(9)

152 2.3.2. Collision kinetics

When grinding, the aim is to maximise the number of successful media collisions. An ideal system will be able to reach the desired particle size distribution for the minimum energy input. Therefore, the best way to increase the number of successful collisions is to increase the probability of a successful collision happening; either by increasing contact frequency or by increasing the average contact energy.

As each simulation was sampled, an approximation for collision frequency 159 is defined as an average of the total number of contacts present per second. In 160 a binary size distribution, there are three types of contact present: 161 contact between two beads of the larger media size, between two 162 beads of the smaller size, or between one larger and one smaller 163 sized bead. This has been quantified for each simulation. As well as 164 frequency, the magnitude and type of collision can be quantified. Beinert et165 al. [52] proposed that a pair of spherical beads in contact would exhibit at 166 least one of six main collision mechanisms, shown diagrammatically in Figure 167 3. Table 1 shows the associated Equations that can be derived from each of 168 these mechanisms. A more detailed explanation can be found in Rhymer et169 al. [40]. The total energy of collision, E_{total} , is the sum of all six components. 170

9

171 2.3.3. Power draw

As the mill was studied at steady state conditions, instantaneous power 172 draw can be found from the summation of all dissipative energies. Most 173 of this energy is media-media collisions, and individual collision values can 174 be found by multiplying the dissipative force components from Equations 3 175 and 4 by the corresponding relative pair velocity (normal and tangential). 176 There are also media-wall collisions and these are calculated similarly. In 177 LIGGGHTS, the relative collision velocity can be found using two methods; 178 either through direct calculation of the damping coefficient, γ , or by using 179 the output data provided to find $\gamma \cdot v$ and then taking the difference between 180 the total and elastic component. This latter method was preferred because it 181 is based on output data rather than inputs, meaning any additional damping 182 assumptions LIGGGHTS uses during simulation to maintain model stability 183 are factored into the overall result. 184

Design equations for power draw within mills also exist. Kwade *et al.* 185 [15] derived a stress model from their studies of different media within a 187 horizontal mill. This was later used by Radziszewski and Allen [53, 54] to 188 create a simple correlation for power that relies on the rotational velocity, ω , 189 perceived dynamic viscosity, η , and shear volume, V_{τ} ,

$$P = \eta \omega^2 V_{\tau}.\tag{10}$$

¹⁹⁰ η is estimated from Gao, Forssberg and Weller's correlations [55]. V_{τ} is a ¹⁹¹ sum of all of the surfaces undergoing shear, I, (Equation 11), with A being ¹⁹² the area undergoing shear, r_s , the radius that the shear is acting, and y, the ¹⁹³ separation between surfaces,

 $V_{\tau} = \sum^{I} A \frac{r_s^2}{y},$ (11)

The Equation proposed by Radziszewski and Allen is very difficult to 194 validate accurately, especially within simulation. η can only be determined 195 through experimental work, and contains several terms to account for the 196 many different parameters within the mill. V_{τ} is also complex to accurately 197 determine as the shear volume can change with rotational velocity. A vortex 198 develops in the centre of the mill, meaning some of the shear surfaces lose 199 contact with the slurry. The previous paper with Osborne [56], also suggested 200 that the surfaces between pins needed to be included in the shear volume to 201 be more robust for a greater number of designs. 202

203 3. Results and Discussion

The results present time-averaged data for ten seconds of simulation, at 205 20 samples per second. This was once the mill had reached the correct veloc-206 ity, and steady state (See Appendix B). The open-source tool Paraview [57] 207 was used to check that the simulations had run properly and if there were any 208 visual differences between different parameters. All visual images were fur-209 ther post-processed in Blender [58]. The LIGGGHTS data was numerically 201 processed and analysed using Python.

211 3.1. Media segregation

As explained in Section 2.3.1, segregation effects are present due to having multiple media sizes. Figure 4 shows examples of this for the different size ratios. As the ratio increases, the rate of segregation and purity of the

two layers increases; which is as expected from segregation theory of other 215 engineering systems [25, 27, 59]. This is true at 50 rpm, (a)-(e), where the 216 input energy from the attritor cannot create enough lift to overcome perco-217 lation effects, so the smaller media fall to the bottom of the mill naturally. 218 At 250 rpm, these effects are still observed but with less effect as the attri-219 tor energy supplied is large enough to lift the smaller media into the larger 220 ones. This is true in all sizes except $2.5 \,\mathrm{mm}$, Figure 4(j). There is an in-221 sufficient volume of smaller media to generate the height that would bring 222 some into the path of the attritor, despite being 80% of the total media 223 count. Industrially, this is significant because a large percolation effect and 224 segregation could result in inefficient grinding if the direction of the slurry 225 flow is incorrect. It has already been discussed that optimal grinding occurs 226 when the bead-to-particle ratio is around 20:1 [13], meaning smaller media 227 should produce a finer grind and should therefore be at the end of the slurry 228 flow if possible for optimal comminution [9, 10]. This would lead to an ideal 229 flow from top to bottom but could lead to channelling and other flow-related 230 issues if not controlled correctly. As gravity would be drawing slurry down-231 wards, residence time would also be reduced by doing this, so it would be 232 necessary to introduce some level or flow control. 233

Figure 5 shows the azimuthally projected (r-z axis) volumetric bead distribution for different attritor speeds and size ratios. As in Figure 4, these were all for a 20% count of 10.0 mm media. Volumetric plots were used to provide representative spatial distributions inside the mill. These show the media segregated by the separation of the different colours. Interestingly, each plot shows a continu-

ous band of equal volume fraction (the white region). The region 240 is larger at higher attritor speeds and smaller media size ratios. 241 As the size ratio increases, this transition becomes smaller and the 242 colours intensify, suggesting stronger segregation is present. At 243 higher speeds, a vortex forms in the centre of the mill because 244 of the increase in centrifugal force [15, 60, 61]. In some industrial 245 cases where high impact is required, this is undesirable and baffling 246 would be used to promote media chaos [53]. 247

248 3.1.1. Lacey mixing index

While occupancy provides a qualitative view of segregation, a mixing 249 index provides a quantitative evaluation. Figure 6 plots the Lacey mixing 250 index for the equiproportional simulation for each size fraction and rotational 251 speed. There is generally a negative correlation in the Lacev index as the 252 size ratio increases. Within each line, there is a fluctuation due to noise from 253 the instantaneous nature of the sampling, but otherwise, the values stay 254 fairly consistent for each simulation. The indices for the 3.75 mm are much 255 lower than the rest of the size ratios, especially at low attritor speed. This 256 indicates that the percolation effect is significant enough for almost complete 257 segregation. However, as the attritor velocity increases, the smaller fractions 258 are lifted by the attritor, creating enough re-circulation to at least partly 259 overcome the segregation effects. 260

A time-averaged value for Lacey can also be calculated and plotted by media count proportion for each simulation (Figure 7). In agreement with Figure 6, these show that the larger the size ratio, the lower the index value, which concurs with previous segregation theory [59]. The value is also highest

when the proportion of 10 mm beads is around 30-40%, though the exact 265 proportionality depends on the size ratio between the two media sizes. This 266 might be because they are closer to equivolume; something which appears 267 to have a greater influence than numeric quantity. While most curves are 268 concave in nature, higher size ratios at low attritor velocities are convex to the 269 abscissa. Industrially, these plots show that uncontrolled size distributions 270 could become a big concern if there are large effects from having multiple 271 sizes as hypothesised. It also potentially reveals that there are limits in size 272 ratio, proportionality, attritor speed and system design that will cause poor 273 re-circulation and will want to be avoided when optimising milling equipment. 274

275 3.2. Media velocity

The average velocity versus 10 mm volume fraction is shown in Figure 8. 276 Each subplot shows the results for a different size ratio. A greater propor-277 tion of 10 mm beads leads to greater velocities. The reason is that contact 278 dissipates energy, so with smaller beads, there are more contacts dissipat-279 ing energy. More of the media can also fit in the dead region previously 280 described. There is an approximately linear relationship between attritor 281 speed and media velocity. That linearity agrees with the results of previous 282 work [40], because the tip speed of the attritor changes linearly for the se-283 lected rotational speeds. As large amounts of smaller media are introduced, 284 the velocity further decreases. As with the changes by proportion, the trend 285 can be described through the additional dissipation of the extra contacts 286 within the mill. At higher attritor speeds, there is media separation because 287 of the centrifugal force generated which also contributes to loss of contact 288 between media. 289

Velocity field maps have been generated to show the 2-D behaviour of the 290 mill (Figure 9). Each simulation presented had an attritor speed of 250 rpm. 291 If the mill is exclusively comprised of small media, Figure 9(a), the volume 292 of the highest velocity is larger when the media are bigger. This goes back 293 to the principles of fewer media contacts so less energy is dissipated. As the 294 size starts to decrease, the regions of the highest velocity are raised towards 295 the upper pins, with less velocity lower down. The tighter packing leads to 296 more collisions and a faster dissipation rate. 297

When media count is equiproportional, (b), there is a counteracting effect, 298 with a smaller second diameter providing greater average velocity in the 299 central milling area. This is because of the segregation effect, meaning if the 300 second media size is small, the area between the pins is almost all 10 mm 301 media. When 7.5 mm media are included, there is a large enough volume 302 and slow enough segregation rate to mean that some media get stuck within 303 the pin region and create extra collisions. However, this is not the case when 304 3.75 mm media are the smaller fraction. The size ratio means that the smaller 305 media percolate and gather at the bottom, meaning the high energy regions 306 are almost exclusively 10 mm media. 307

308 3.3. Media force

While velocity has a non-linear correlation with media proportion, the average media force shows linearity, especially when considered by count proportion. Figure 10 shows the average media force plotted against proportion for different size ratios. These plots show that as the proportion of 10 mm beads increases, the net force on each media bead also increases [4, 19]. This is expected as larger media carry larger mass providing density

is kept constant. However, the fact that the distribution is linear when the
average force is plotted against the count proportion is surprising. While it
would be natural to assume a weighted proportion based on two individual
media forces that stay reasonably constant, it would imply that there is a
uniform behaviour throughout.

The linear plots in Figure 10 were combined and normalised by the tip 320 speed to consider the effect of increased attritor velocity, shown in Figure 11. 321 Each plane does not sit perfectly on one another, particularly as the propor-322 tion of 10 mm beads and smaller bead sizes increase. This aligns with the 323 conclusions of previous work [40], which suggested that increased attritor ve-324 locity provides a greater average force per unit of input energy. In the case of 325 different media sizes and proportions, the average media force increases with 326 a greater proportion of large media. Again, this can be expected but there is 327 less linearisation as media diameter changes. Here, there is a slightly expo-328 nential trend as media diameter increases, particularly at higher proportions 329 of smaller media. 330

331 3.4. Contact energies

332 3.4.1. Energy distribution

Successful grinding depends on collisions between media particles containing sufficient energy to fracture any particulate material which could be trapped in between them. As one of the limitations of the simulation method is the ability to include any particulate material or breakage models, there needs to be some way of predicting the grinding potential. Linear distribution plots showing the spread of collision energies within the mill is one way of doing this. These generate a probability distribution of the likelihood of

a successful collision. This data has been collated overall time outputs using
the Beinert *et al.* [52] calculation method (See Section 2.3.2).

Figure 12 shows different energy distributions when 7.5 mm are the smaller 342 media. As the attritor velocity increases from $50 \,\mathrm{rpm}$ in (a), to $200 \,\mathrm{rpm}$ in 343 (d), the energy distribution shifts to the right, meaning that on average, each 344 collision has more energy and is more likely to cause grinding. This is ex-345 pected because more energy is being supplied by the faster attritor. However, 346 as the proportionality of 7.5 mm media increases, the mean energy per colli-347 sion decreases because smaller media have a lower individual mass. There is 348 also an increase in collision frequency as the proportion of 7.5 mm beads 349 increases because total media volume was kept constant, meaning 350 there a greater number of media and an increase in total surface 351 area. 352

If the trends are shown for different size ratios, Figure 13, the same pat-353 terns are observed but become more extreme as the size ratio increases. By 354 the time the smaller media is reduced to $3.75 \,\mathrm{mm}$ in Figure 13(d), a clear 355 bimodal peak can be observed from the energies supplied by the different 356 collision types. This can be validated in Figure 14, which splits each line 357 into the individual collision types. There is a clear peak for contacts between 358 two large, and two small media because of the mass differences, and the seg-359 regation which has been previously commented on, meaning that those mass 360 fractions interact far more discretely than when the ratio is smaller. The 361 contacts between a 10 mm and one of the smaller sizes span a wide range 362 of energies compared to the ones for the single-size contacts. Within an in-363 dustrial setting, this is very relevant because if media wear is causing size 364

reduction and segregation of the media, shifting the energy curves to the left
and reducing the probability of high energy, successful collisions and grinding
potential.

Figures 12 and 13 have shown that there could be some compromise be-368 tween having enough high-energy collisions from the large media and having 369 a high frequency of collisions from the smaller media. Therefore the combined 370 energy for each simulation was plotted in Figure 15, with different sections 371 of each bar representing the individual collision types. The total collision 372 frequency is also shown for reference (Figure 16). The increased number 373 of contacts from the smaller media, particularly in the 3.75 mm case, (d), 374 shows that there is a clear compromise point where the amount of grinding 375 energy produced is higher. This is because there are more frequent collisions 376 across the mill from the smaller size, but the quantity of large media is still 377 high enough. There is also a significant contribution from impacts between a 378 large and a small bead, suggesting the boundary is high enough to be within 379 the pin region. Industrially this is highly significant because the media size 380 could be controlled and topped up to get specific age and size distributions 381 of the media and optimise energy transfer. Different sizes can also be added 382 to the virgin media to aid this. Wear profiles and correlations can be 383 generated [62-65], with Altun et al. [20] being one such example 384 in the mining industry. They correlated energy input to mass loss, 385 meaning a rough size distribution can be found. The initial size 386 going into the mill can also be controlled and does not have to be 387 a single size, aiding a polydisperse size distribution.

389 3.4.2. Energy distribution by height

The energy distributions were also plotted by height to show if there 390 are any vertical differences because of the media segregation (Figure 17). 391 The peak energy is just below the middle pair of pins for all simulations 392 and then has a skewed Gaussian distribution. This makes sense because the 393 middle pins are where the greatest energy input density is. For most cases, 394 energy from contacts between 10 mm media dominates. It is only when high 395 proportions of 7.5 mm and 5.0 mm media are used that the contribution of 396 the smaller sizes dominates. In these cases, they are volumetrically dominant 397 and responsible for the majority of contacts. 398

There is more energy below the pins than above as more media are at 399 the bottom due to gravity. Even though it was identified as a lower velocity 400 zone, the bottom 10 mm can contribute as much as $60 \,\mathrm{J \, s^{-1}}$. As the size ratio 401 increases, more contacts can fit within each cell and the bottom layer of 402 media becomes smaller in average depth, reducing boundary effects. Within 403 a mill, a layer of media cage at the outer surfaces, often having much slower 404 kinetics than those slightly further inwards. If the media diameter is reduced, 405 this layer becomes smaller. Therefore the perceived active region increases. 406

To draw a more direct comparison, the total energy by height has also been plotted linearly (Figure 18). At high size ratios, the profiles are very similar because the mill is mostly 10 mm media by volume, particularly within the pin region. At smaller size ratios, there are slight differences. Low proportions of 10 mm media produce a higher peak and greater total area. This agrees with the observations seen in Figure 17, where a 0.1 fraction of 10 mm media gives a slightly higher energy. At high proportions of 3.75 mm

media, there is a significant rise in the peak stress energies observed. This 414 is because there is still a large volume of $10 \,\mathrm{mm}$ beads, but enough $3.75 \,\mathrm{mm}$ 415 beads to fill the spaces between them effectively. At large proportions of 416 10 mm media, the spaces between media and small gaps between attritor 417 and vessel mean that the media are quite spaced and unable to transfer 418 their momentum fully. Improved packing from a mix of sizes allows for 419 more effective momentum transfer. These results give a good indication of 420 optimal media distribution. A smaller base size with a handful of slightly 421 larger media provides a strong result, suggesting that including multiple bead 422 sizes increases the optimal grinding performance. 423

While overall stress energy contribution is important, milling relies on in-424 dividual contacts having sufficient energy to reduce particle size successfully. 425 Figure 19 and 20 show for each size ratio, the contact frequency and stress 426 energy per contact. While Figure 18 shows a setup with balanced volumes 427 of a higher size ratio as a potentially optimal choice, the energy per contact 428 is much lower throughout the milling space. As explained, the smaller me-429 dia carry much lower momentum, so transfer less energy through individual 430 collisions. What is causing the large amounts of energy transfer is the much 431 larger quantity of contacts recorded, particularly in the lower part of the mill 432 where most of the volume is smaller media. For the most part, 10 mm media 433 are present in large quantities and the number of contacts is similar at the 434 top of the vessel because the segregation effects mean that the upper region 435 is almost solely larger media. This changes very little until the smaller media 436 are added in large quantities, especially when the size ratio increases. 437

438

This balance between high energy and high contact frequency is critical

when designing a milling process, as it gives operators a very controllable 439 way of optimising the overall energy distributions observed within the mill 440 and can be tailored for specific materials and particle size specifications. The 441 locations and size of the contacts also give further evidence that the flow of 442 material in continuous processes should be from top to bottom so that finer 443 particles can interact more with smaller media which produce a finer grind 444 [9, 10]. This is not always the case, with many vertical mills preferring a 445 pump-up approach as it maximised slurry residence time [66, 67]. 446

447 3.5. Power draw

While most of the previous analysis has centred around the theoretical 448 effectiveness of milling, the power draw can be used to determine the over-449 all efficiency. The power has been calculated using the method described in 450 Section 2.3.3, which assumes that the power draw equals the energy dissi-451 pated in each contact. The result for each simulation is shown in Figure 21. 452 There are slight deviations in results because of the instantaneous nature of 453 the sampling, but for each ratio, there are some common themes. In most 454 cases, the power draw falls slightly as the smaller size is added until media 455 quantities become roughly equiproportional. Then, as the smaller size be-456 comes the dominant fraction the power increases and by the time the mill 457 becomes exclusively smaller media, the power is slightly greater than when 458 the mill is full of 10 mm media. There are more contacts when the mill con-459 tains smaller media, so there are more dissipative events. This is similar in 460 most cases, apart from the $3.75 \,\mathrm{mm}$ size fraction, Figure 21(d). There is a 461 more significant power decrease at a much higher numerical proportionality. 462 This is because at numerical ratios of 0.8-0.9, the $3.75\,\mathrm{mm}$ media mostly sit 463

below the attritor pins, while in the attritor region, there are fewer of the 464 10 mm media to dissipate energy, than compared to a lower proportionality. 465 However, the trend continues even when all of the media are of a smaller 466 size. By this point, the media are small enough to behave more like a fluid 467 body rather than as individual elements. This means the attritor has a lower 468 profile when moving through the media, and the gaps between the attritor 469 and vessel are large enough to fit several media. This is highly significant 470 industrially, as reduced power draw and favourable collision kinetics suggest 471 that a mixture of small and large media might be better than having a single 472 media size. 473

As seen in previous studies [36, 40], there is a non-linear trend as the 474 attritor velocity increases, which is to be expected as more energy is supplied. 475 Radzizewski and Allen [53, 54] concluded that the power draw is proportional 476 to the rotational velocity squared (Equation 10). The plots show a reasonable 477 level of agreement with this, especially when considering that the power is 478 also related to the shear volume, and this changes with velocity because 479 different surfaces come into and out of contact with the media, hence why 480 there is an inexact correlation to the squared velocity. 481

For each simulation, the stress energy was divided by the power draw (Figure 22). This shows that in most cases, the energies observed are proportional to the power supplied, meaning that an increase in attritor velocity linearly increases the energy generated. There is a lower efficiency at 50 rpm, meaning that the contacts are much lower in energy than the power used. Given that from Figure 21 the trends are non-linear, the stress energies are very low compared to higher attritor speeds. There is a special case where

the efficiency is much higher. This is when the size ratio is high but there is a high quantity of smaller media. This is because the energy is maintained within the contacts and less is dissipated which is the calculation assumption. The smaller media are easier for the attritor to move through, reducing potential resistance and increasing single-body motion.

494 3.6. Full polydispersity

Three final simulations were run to see if the observed effects of the binary 495 system also occurred when a fully polydisperse set of media was used. The 496 first setup used an equal quantity of all six media sizes at 50 rpm, with an 497 equivalent total volume to all prior simulations. The second was to use the 498 same setup but at 250 rpm. The final one used equal volumes of each fraction, 499 again filled to the same overall volume. All three were run for 60 seconds 500 from the point of the first attritor movement, just like the steady state checks 501 (Appendix B). 502

Figure 23 and Video 2 shows the progression of the simulation containing 503 the same number of media at each size fraction. The media segregate as 504 expected, with the larger fractions moving towards the free surface at the 505 top. This is especially prevalent for the equal quantities simulation because 506 most of the volume comprises the larger media. At the start, the small media 507 percolate as they can move through the vortices formed from the other media 508 packing together without the driving energy of the attritor. This is why they 509 start at a lower centre of mass than other fractions, and fall towards their 510 stable level very quickly. The larger media cannot do this and require other 511 driving factors to move axially. This is why they take longer to reach a 512 stable centre of mass and even after this has been reached, there is still a 513

514 small amount of fluctuation.

As the velocity increases to 250 rpm, Figure 24 and Video 3, the same 515 effect of segregation is seen. However, all of the media fractions sit higher 516 than at 50 rpm, and this is due to the formation of the vortex in the centre 517 of the mill. A convective remixing effect that is able to lift all media, but 518 especially the smaller fractions sitting within the bottom section. The reason 519 for the delayed response in the small fractions is that the mill was accelerated 520 gradually, meaning the vortex took time to form initially. It forms from the 521 free surface and then develops further down the vessel. This means the same 522 initial percolation effect occurs, like in Figure 23, before convective remixing 523 dominates and elevates the smaller media. The centrifugal force of the cir-524 cular motion also means that media want to push outwards, and eventually 525 upwards because they have nowhere else to move. This is especially true 526 at the free surface because media are not obstructed by anything above, so 527 elevate towards the upper section. 528

In contrast, Figure 25 and Video 4 show the result of putting all six me-529 dia fractions in the mill with equivolume quantities. The full polydisperse 530 nature of the simulation means that a densely packed bed is formed by the 531 media, and at 50 rpm, the energy supplied is not enough to disturb the bot-532 tom layers that sit under the attritor. This is bad for grinding performance 533 because energy is dissipated quickly, and is not transferring to all of the 534 beads successfully, reducing the active milling region. Above the dead zone 535 at the bottom, the media is largely segregated as seen previously but looks 536 less separated on the centre of mass by time plot because of the effects of 537 the lower region. There is still percolation from the small media but less of 538

⁵³⁹ it because of the better media packing from the system.

A comparison of the contact energies for three cases is shown in Figure 540 26. The polydisperse case setup in Figure 24 falls somewhere in between the 541 results of an equal count, low size ratio $(7.5 \,\mathrm{mm} \,\mathrm{media}, 50\%)$ and a high 542 size ratio with significant count of small media (3.75 mm, 90% small) charge. 543 Because of the segregation effects, the size ratios in the polydisperse case 544 locally are quite low for the most part, but there will still be contacts between 545 media of a high size ratio. Because there are three types of contact, the 546 energy distribution, Figure 26(a), is much broader than in the other cases and 547 does not show distinctive peaks like the high size ratio simulations do. The 548 remaining distributions fall roughly in line with other setups, meaning the 549 mill is performing consistently. It can also be seen that when plotting against 550 height, the polydisperse case performs almost identically to the low size ratio 551 case within the top $50 \,\mathrm{mm}$ of the vessel. This is because the compositions 552 are likely to be very similar in that it will mostly be 10 mm media with some 553 7.5 mm also present. 554

Industrially, vertical mills containing polydisperse media are the most 555 likely to occur because of how they are operated and the natural ageing ef-556 fects caused by bead wear. However, there is a possibility for this to be 557 controlled more strictly to obtain a specific distribution that suits the ap-558 plication. Mineral processing for instance relies on high-impact contacts to 559 break hard particles. Meanwhile, the food industry, where particles are softer, 560 may utilise other size ratios and proportions with lower peak energies but a 561 greater contribution due to the volume of contacts. As well as the observed 562 effects, this research also highlights why simulations are becoming more rel-563

⁵⁶⁴ evant within research because they can very easily show that there can be
⁵⁶⁵ inefficiencies within the mill and where it is within the equipment. This can
⁵⁶⁶ lead to better designs of both the attritor and/or vessel to reduce or remove
⁵⁶⁷ this effect.

568 4. Validity

From previous work [36, 40, 41], it was believed that the mod-569 elling parameters were most representative to characterise the mo-570 tion of the media and inter-particle stresses. The aim was to show 571 the effects of changing media size fractions in isolation, rather than 572 try to replicate a specific industrial example which would cost thou-573 sands to validate experimentally. Therefore, the study was re-run 574 with a restitution coefficient of 0.1 and friction coefficient of 0.9. 575 Using low restitution and high friction was the opposite of the 576 initial investigation, and qualitative agreement will show that the 577 observations are independent of the other values used and can be 578 considered valid, as all alternative combinations will fall in between 579 the two cases demonstrated. 580

While there are some differences in the 3.75 mm case, the Lacey index in most cases remains largely unchanged compared to the original (Figure 7), particularly at higher speeds (Figure 27). The high friction increases the Lacey index at lower speeds due to the resistance of some of the percolation drivers. When the size ratio is smaller, Figure 27(a), the effects are similar to the original plot because the reduced segregation in these simulations is less affected ⁵⁸⁸ by the change of resistance.

One of the major findings was that there is a compromise be-589 tween lots of contacts between smaller media and some high-impact 590 collisions from the larger beads. If this plot is presented again with 591 the new parameters, as shown in Figure 28, the results qualitatively 592 suggest the same trend as in Figure 15. However, there is a slight 593 shift in where the peak energy is as a proportion due to the changes 594 in the Lacey index previously discussed. In the 3.75 mm simulations, 595 the peak shifts from a large media fraction of 0.1 to 0.3 but the 596 same information can otherwise be extracted in the same way as 597 the original parameters. There is a compromise between having 598 many high-impact contacts and having a high quantity of contacts 599 which qualitatively agrees with the initial batches. 600

The results of Figure 27 and 28 show that the main qualitative findings are independent of specific media properties and therefore can be deemed valid for this trial. The only thing which is changing is the magnitude of the values and optimal population, although neither varies significantly from the original conditions,

606 5. Conclusion

The paper has detailed the effects of changing the proportion and size ratio of a binary dispersed set of grinding media within a vertical stirred mill. Overall, three main conclusions can be drawn:

Including a mix of large and small grinding media increases
 the total stress energy observed, as both high-impact collisions

from the large media and a high contact frequency from lots
of small media are present.

2. Having a high proportion of small media reduces the power
draw by reducing the contact distance between media. This
might improve the overall energy efficiency but we need to
understand the grinding effect. This is proposed for future
study.

3. Segregation of different media sizes is advantageous, as it creates different collision types within the mill. Larger media
produce higher energy contacts to break larger particles while
smaller media can produce a finer grind, so if the slurry flow
can be directed from the top of the mill to the bottom, it
maximises the effectiveness throughout the system.

The scope of the study has been limited to changing the me-625 dia size so that the effects observed could be isolated to a single 626 variable. To demonstrate the validity of the results, the inves-627 tigation was repeated using highly contrasting media properties 628 and qualitatively, they agree with the initial study (See Section 4). 629 While we understand that operators control the mill using different 630 measures to the ones analysed, what we are doing with the simu-631 lation approach is predicting the parameters which would provide 632 strong grinding potential. This reduces the number of physical 633 trials which would then be required to reach an optimal set of op-634 erating conditions because there is a greater understanding of the 635

contact dynamics before online operation. Further investigations
looking at shape and surface property variation between media,
and wear of both media and attritor as a function of time would
have increased the reality of the model. However, these would
also have increased the complexity of modelling and analysing the
effects of changing multiple parameters simultaneously.

642 Declaration of Competing Interest

The authors declare upon submission that there are no competing interests.

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654 Nomenclature

			C.
lon	nenclati	ıre	
	A	Shear area	m^2
	E	Young's Modulus	Pa
	E^*	Effective Young's Modulus	Pa
	F_{ij}^n	Normal contact force	Ν
	F_{ij}^t	Tangential contact force	Ν
	G	Shear Modulus	Pa
	G^*	Effective Shear Modulus	Pa
	g	Gravitational acceleration	${ m ms^{-2}}$
	Ι	Moment of inertia	${ m kgm^2}$
	K^n	Normal stiffness	$\mathrm{Nm^{-1}}$
	K^t	Tangential stiffness	$ m Nm^{-1}$
	m	Particle mass	kg
	m^*	Effective particle mass	kg
	P_n	nth Percentile	-
	R	Particle radius	m
	R^*	Effective particle radius	m
	r_s	Shear radius	m
	S^n	DEM contact law constant	<u> </u>
	t	Timestep	s
	V_{τ}	Shear volume	m^3
	v_{ij}^n	Relative normal velocity	${ m ms^{-1}}$
	v_{ij}^t	Relative tangential velocity	${ m ms^{-1}}$
	β	DEM contact law constant	-
	γ^n	Normal damping coefficient	$ m Nsm^{-1}$
	γ^t	Tangential damping coefficient	$ m Nsm^{-1}$
	δ_{ij}^n	Normal particle overlap	m
	δ_{ij}^t	Tangential particle overlap	m
	ε	Coefficient of restitution	-
	η	Dynamic viscosity	Pas
	ν	Poisson ratio	-
	ρ	Particle density	${ m kgm^{-3}}$
	$ au_{ij}^t$	Rolling friction coefficient	-
	ω	Rotational velocity	$ m rads^{-1}$
4			
		90	
		30	

655 Appendix A: Additional Equations

From Section 2.1, the normal and tangential forces between a pair of colliding particles, F_{ij}^n and F_{ij}^t respectively, were defined. [38, 40]

$$F_{ij}^n = K^n \cdot \delta_{ij}^n - \gamma^n \cdot v_{ij}^n, \tag{3}$$

$$F_{ij}^t = K^t \cdot \delta_{ij}^t - \gamma^t \cdot v_{ij}^t.$$
(4)

For each equation, the total force is made up of the difference between the elastic and dissipative components in that direction. Starting with the normal force (Equation 3), the elastic component is found by multiplying the normal overlap between the particles, δ_{ij}^n by the stiffness constant, K^n , which is found in Equation 12,

$$K^n = \frac{4}{3} E^* \sqrt{R^* \delta^n_{ij}}.$$
(12)

 E^* is the effective Young's modulus between particles, with E_i and E_j being the individual moudii.

$$E^* = \left(\frac{1-\nu_i^2}{E_i} + \frac{1-\nu_j^2}{E_j}\right)^{-1},$$
(13)

and R^* is the effective radius,

$$R^* = \frac{R_i R_j}{R_i + R_j}.$$
(14)

⁶⁶⁶ ν is the Poisson ratio of each particle. The dissipative component of Equation ⁶⁶⁷ 3 is calculated from the relative normal velocity of the particles v_{ij}^n , which is ⁶⁶⁸ multiplied by the dissipative coefficient, γ^n .

$$\gamma^n = -2\sqrt{\frac{5}{6}}\beta\sqrt{S^nm^*} \cdot v_{ij}^n.$$

 S^n is described in Equation 16 [68],

$$S^n = 2E^* \sqrt{R^* \delta_{ij}^n},\tag{16}$$

(15)

 $_{670}$ β is a proportionality constant, with ϵ being the particle restitution coeffi- $_{671}$ cient,

$$\beta = \frac{\ln(\varepsilon)}{\sqrt{\varepsilon^2 + \pi^2}}.$$
(17)

and lastly, m^* is the effective particle mass,

$$m^* = \frac{m_i m_j}{m_i + m_j}.$$
(18)

The tangential force equation (Equation 4), follows the same basic philosophy as for the normal force. This time, δ_{ij}^t represents the tangential overlap and this is multiplied by the tangential elastic coefficient to get the elastic component of the collision force,

$$K^t = 8G^* \sqrt{R^* \delta_{ij}^n}.$$
 (19)

 G^* is the effective shear modulus and is calculated in a similar way to the effective Young's modulus,

$$G^* = \left(\frac{1-\nu_i^2}{G_i} + \frac{1-\nu_j^2}{G_j}\right)^{-1}.$$
 (20)

To calculate the dissipative component, the relative tangential velocity, v_{ij}^t , is multiplied by the tangential stiffness constant, K^t ,

$$\gamma^t = -2\sqrt{\frac{5}{6}}\beta\sqrt{K^tm^*}\cdot v_{ij}^t.$$

(21)

Equations 3 and 4 form the basis for calculating power draw because the normal and tangential forces for each contact are provided in the LIGGGHTS output data. It is assumed that the power draw is equivalent to the energy dissipated in the system.

685 Appendix B: Steady state checking

Because the grinding media were inserted into the mill randomly and then allowed to move and segregate freely, a check had to be done to ensure a comparable state had been reached for all simulations. This was done by running three additional simulations using the worst mixing conditions. These are stated:

- 10 mm vs 7.5 mm particles (smallest media ratio)
- equal proportionality (zero proportionality bias)
- 50 rpm rotational speed (slowest attritor rotation)

The progression of these simulations is shown in Figure 29 and Video 1. 694 The grinding media were seeded in different ways: all of the 10 mm media 695 first, (a), randomly, (b), and all of the 7.5 mm beads first, (c). The reason 696 was that barring equipment inefficiency, all three should naturally converge 697 to a similar state due to entropy, and the time taken would be useful in 698 determining whether the ten-second period stated was long enough for the 699 majority of simulations to normalise. Overall, the three simulations tend 700 towards a similar state and except for a handful of media, almost completely 701 segregate. Figure 30 shows the progression numerically by taking the vertical 702 centre of mass for each media fraction. For each case, the fractions segre-703 gated, and eventually stabilised to within a constant level. However, they do 704 not all align perfectly because of the dead zone at the bottom of the mill, 705 as stated in Section 5. This section largely resembles its initial fill state and 706 fractionally adjusts the overall averages seen. Even at 60 seconds, Figure 707

⁷⁰⁸ 29 shows that the bottom layer of media has been largely unaffected while ⁷⁰⁹ everything above has almost completely segregated. It is also unsurprising ⁷¹⁰ that in the instance where small media were filled first, this is the first case ⁷¹¹ to reach a steady state as the media are already naturally segregated.

What Figure 29, 30 and Video 1 prove is that the ten seconds analysed is 712 appropriate, as despite the mills having not reached full stability in the cases 713 presented, the changes by the end are fairly insignificant from the final result. 714 This is also the worst-case assumption, so other simulations are expected to 715 segregate faster. The centre of mass for the mixed fill is less than 2 mm 716 difference from the full stability of the simulation so further changes are 717 minimal. Therefore, the period suggested for analysis (shaded in Figure 30) 718 has been deemed appropriate for all simulations. 719

To further prove that the simulations have reached a steady state, the seg-720 regation progression of the $7.5 \,\mathrm{mm}$ simulations at 50 rpm is shown (Figure 721 30). These show that the average z-position starts from the same point and 722 then spreads as the two fractions segregate. Higher proportions of 7.5 mm 723 media push the average height of both fractions up because of the greater 724 volume of smaller beads. Most importantly, the simulations reach an approx-725 imate steady state by the analysis period, allowing for some natural fluctu-726 ation. This further proves that the period for further analysis is appropriate 727 and that the starting point should be after 45 seconds. 728

729 Appendix C: Sub-region analysis

Another way to check for segregation is to consider media within specific 730 sub-regions of the mill. Three cases were run, shown in Figure 31, with 731 red sections denoting areas not considered in a particular set of analyses. 732 The sub-region only considering the pins, Figure 31(b), was because this 733 is theoretically where the highest energy media should be because the pins 734 supply the energy to the mill. Meanwhile, Figure 31(c) did not consider 735 a region previously identified as containing media with significantly lower 736 kinetics than the rest of the design [40]. 737

As Figure 30 showed, there is significantly less movement at the bottom 738 of the vessel because it is below the bottom of the attritor. If this region 739 is removed, as represented in Figure 31(c), the fraction of media which re-740 mains changes significantly in some cases (Figure 32). In almost all cases, 741 a minimum of 20% of the media total sits within this dead zone. However, 742 the worst case is when 40% of $3.75 \,\mathrm{mm}$ media are included, where this value 743 drops to just 40% of media sitting above the bottom 20 mm of the vessel 744 height. This is bad design that such a large proportion of media sit within a 745 sub-optimal grinding region. 746

Figure 33 plots the same proportionality but for the media in the pin region. These charts show what fraction of the media are in the most effective region of the mill for energy transfer. The proportion of media in line with the pins varies based on velocity and size ratio. As you add smaller media, they segregate toward the bottom of the mill, and with more being able to fit into the same volume as the larger size, reduce the number within the pin region. The greater attrition coming from smaller media [4] will

also cause more wear on the lower attritor pins, as demonstrated by 754 Altun et al. [20] who observed 2.5 times the amount of wear on the 755 bottom pins of a HIGMill compared to the top ones. This becomes 756 more extreme when higher size ratios are considered because the smaller size 757 takes up a lower overall volume despite being kept in constant numerical 758 proportions. Increasing the rotational velocity also reduces the proportion 759 between the pins. However, this is due to the increased centrifugal force 760 pushing media into the region above the pins. 761

Within Figure 33(d), there is a linear trend at low rotational velocity and high 10 mm media proportionality. This is because almost all of the 3.75 mm media percolate to the bottom, as seen in Figure 32(d), and take up such a small amount of the total fill volume that the fraction observed is very close to the proportion of 10 mm media less a constant volume above and below the pins.

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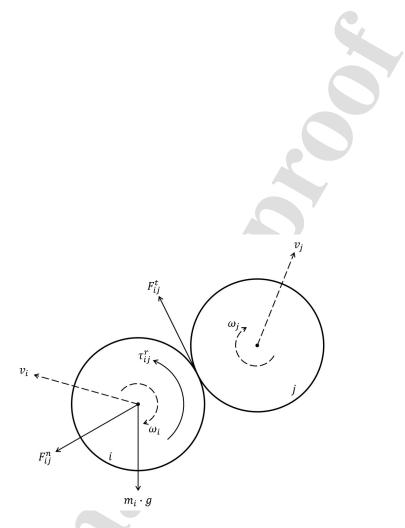


Figure 1: The resultant forces and torques of two particles, i and j, each of radius, r, in contact with each other [40]

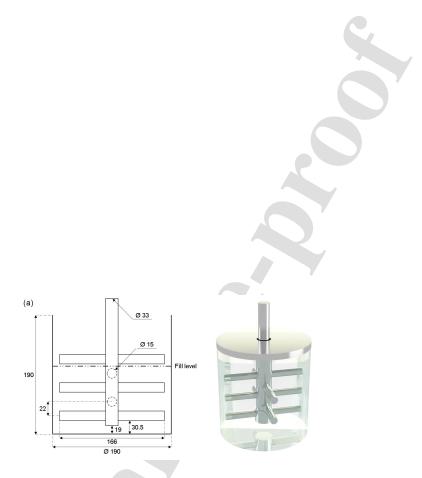


Figure 2: (a) Schematic representation of the mill with all key dimensions marked in millimetres. (b) 3D CAD model of the mill used [40].

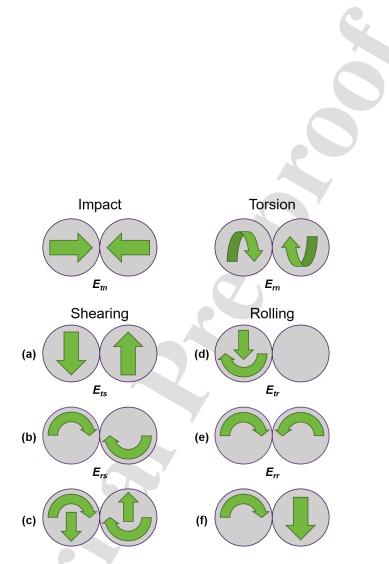


Figure 3: The eight collision mechanisms presented by Beinert *et al.* [52], separated into impact, torsion, shearing ((a)-(c)) and rolling ((d)-(f)) effects [40].



Figure 4: Images showing the state of 80% by quantity of different size media beads after 10 s of milling. (a) & (i) 7.5 mm, (b) & (g) 6.25 mm, (c) & (h) 5.0 mm, (d) & (i) 3.75 mm and (e) & (j) 2.5 mm. (a)-(e) are at 50 rpm, while (f)-(j) are at 250 rpm.

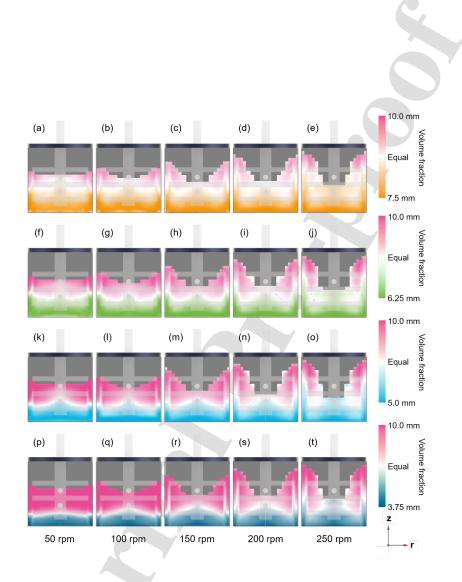


Figure 5: Time averaged volume occupancy fraction plots for an 80% count of the smaller media fraction. (a)-(e) 7.5 mm, (f)-(j) 6.25 mm, (k)-(o) 5.0 mm and (p)-(t)
3.75 mm. The rotational velocity increases from 50 rpm on the left to 250 rpm on the right of each row. Dark grey areas are empty.

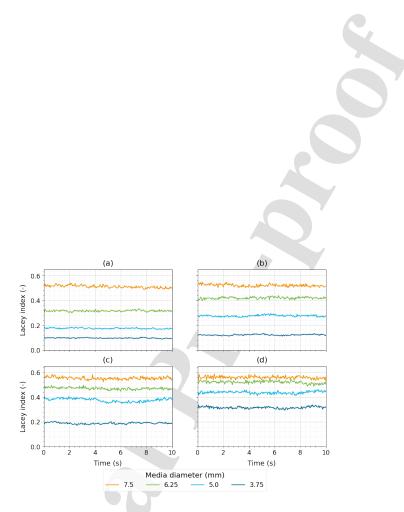


Figure 6: Progression of Lacey mixing index values over time for different simulations. (a) 50 rpm, (b) 100 rpm, (c) 150 rpm, (d) 200 rpm.



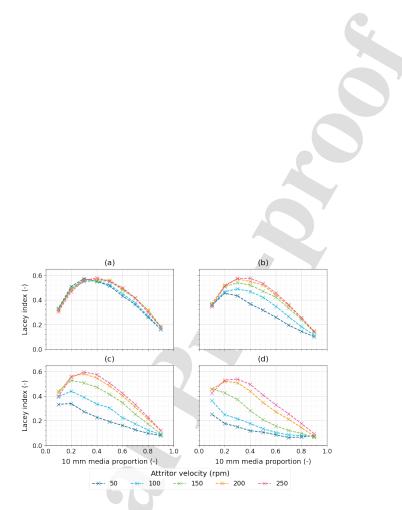


Figure 7: Calculated Lacey mixing indices for each simulation, plotted against count proportion. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, and (d) 3.75 mm.



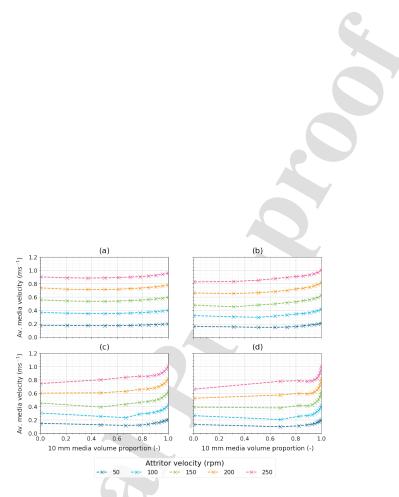


Figure 8: Plots showing the average media velocity at different attritor velocities for different volumetric media proportions. Smaller size media: (a) 7.5 mm, (b) 6.25 mm (c) 5.0 mm, (d) 3.75 mm.

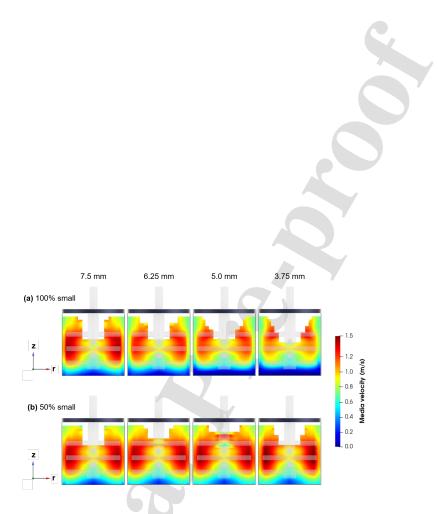


Figure 9: θ -averaged velocity field maps of the mill for different media proportions and size ratios at 250 rpm. (a) 100% small media, (b) 50% small media.



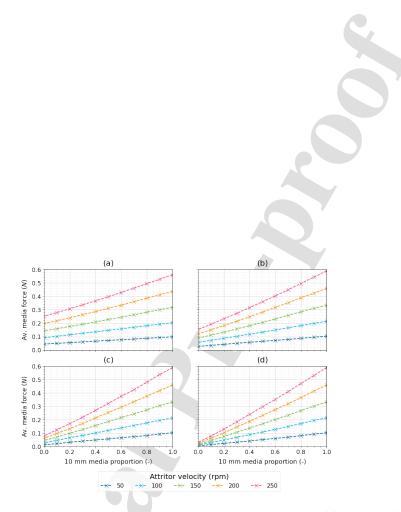


Figure 10: Linear force plots by media count proportion and velocity. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, (d) 3.75 mm.



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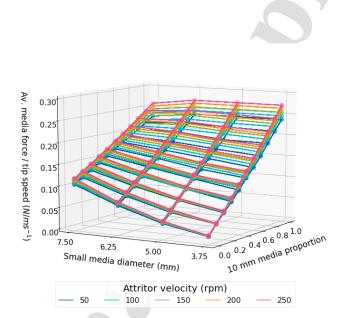


Figure 11: 3D planes showing the average media force, normalised by attritor tip speed for different media sizes and proportions.

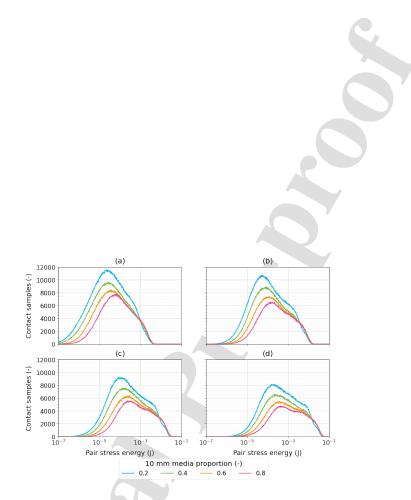


Figure 12: Overall collision energy spectra for different proportions of 10 mm vs 7.5 mm beads at different rotational velocities. (a) 50 rpm, (b) 100 rpm, (c) 150 rpm, (d) 200

rpm.

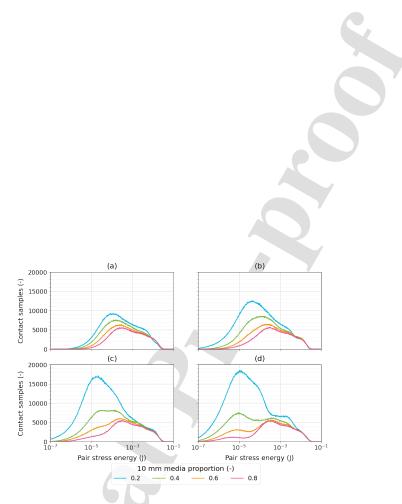


Figure 13: Overall collision energy spectra for different fractions of media beads at 150 rpm with different small sizes used. (a) 7.50 mm, (b) 6.25 mm, (c) 5.00 mm, (d) 3.75 mm



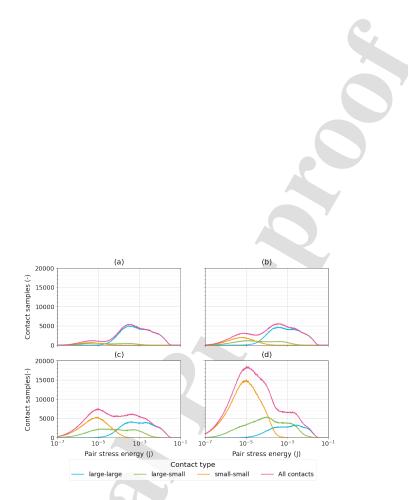


Figure 14: Collision energy spectra by collision type at different size proportionalities at 150 rpm (Breakdown of Figure 13(d)). (a) 20% small media, (b) 40% small media, (c) 60% small media, (d) 80% small media. All plots are for mills run at 200 rpm.



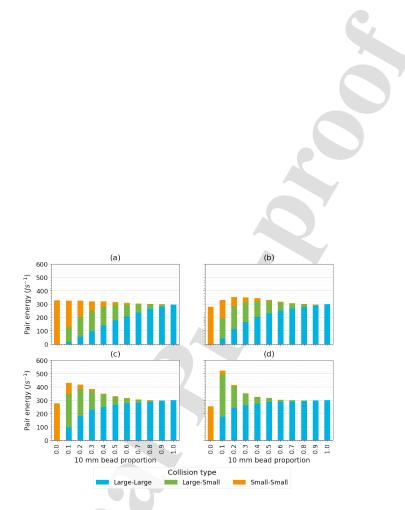


Figure 15: Overall collision energy spectra for different bead proportions and ratios at 250 rpm. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm and (d) 3.75 mm.

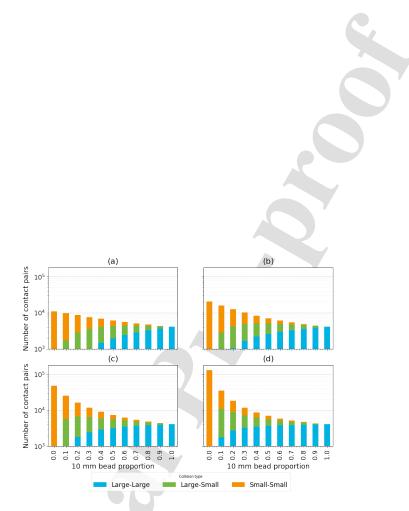


Figure 16: Average number of contact pairs for different bead proportions and ratios at 250 rpm. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm and (d) 3.75 mm.



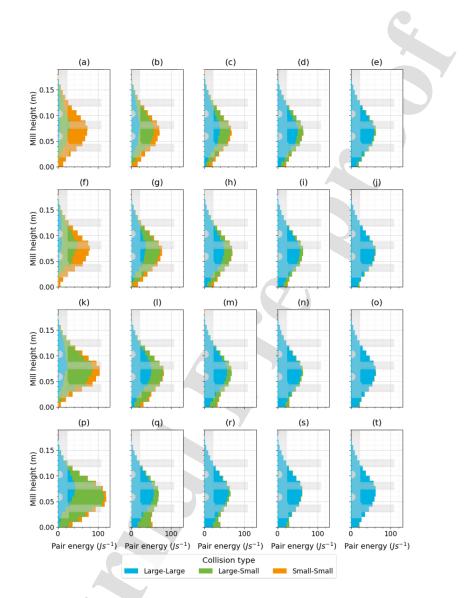


Figure 17: Pair stress energies by contact type at different heights of the mil, arranged by media diameter and proportionality. Left to right: 10 mm media at 10%, 30%, 50%, 70% and 90% proportionality. Top to bottom: smaller size of 7.5 mm, 6.25 mm, 5.0 mm and 3.75 mm. All plots are at 250 rpm.

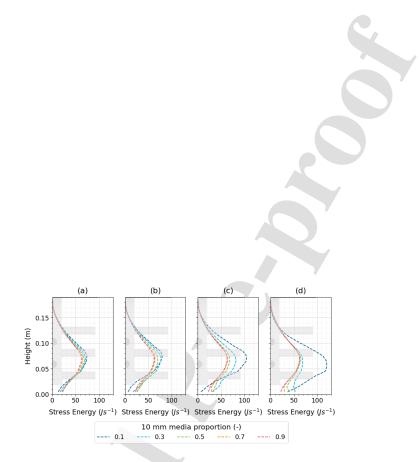


Figure 18: Distribution of pair stress energies per second by height for different proportionality at different media sizes. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, and (d)

 $3.75\,\mathrm{mm}.$

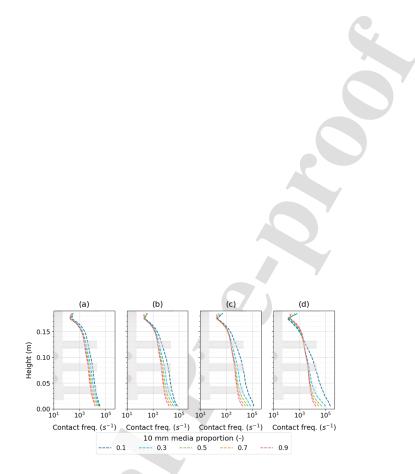


Figure 19: Contact frequency by height for different proportionality at different media sizes. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, and (d) 3.75 mm.

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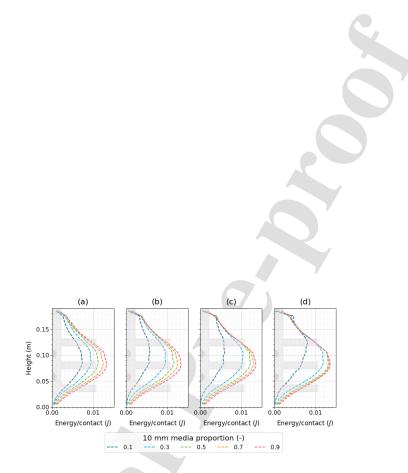


Figure 20: Energy per contact at different heights for different media sizes. (a) 7.5 mm, (b) $6.25 \mathrm{mm}$, (c) $5.0 \mathrm{mm}$, and (d) $3.75 \mathrm{mm}$.



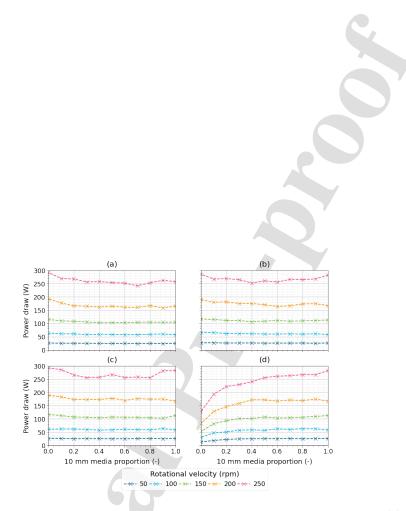


Figure 21: Calculated power draw for each simulation at different media ratios. (a) $7.5\,\rm{mm},$ (b) $6.25\,\rm{mm},$ (c) $5.0\,\rm{mm},$ and (d) $3.75\,\rm{mm}.$



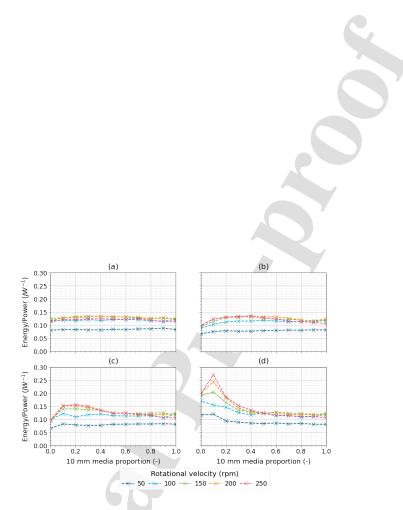


Figure 22: Efficiency plot of pair energy (Figure 15) divided by power draw (Figure 21) for each simulation. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, and (d) 3.75 mm.



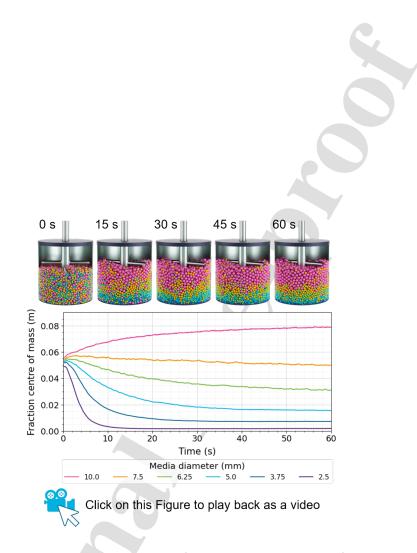


Figure 23: Image showing the progression of an equal count simulation of all six media diameters at different times, agitated at 50 rpm, with an additional plot showing the vertical centre of mass by media fraction.

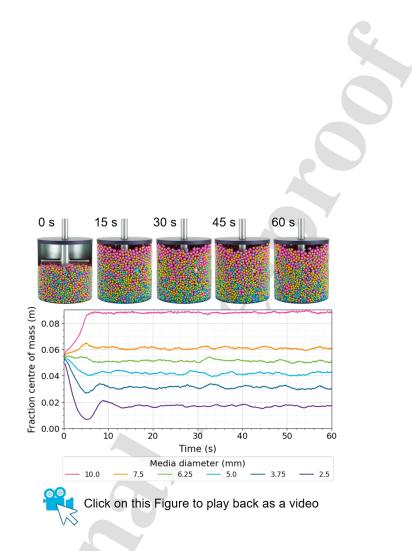


Figure 24: Images showing the progression of an equal count simulation of all six media diameters at 250 rpm for different times, with an additional plot showing the vertical centre of mass by media fraction.

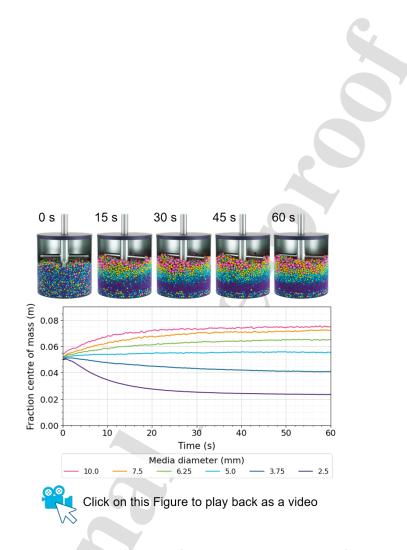


Figure 25: Images showing the progression of an equal volume simulation of all six media diameters at 50 rpm for different times, with an additional plot showing the vertical centre of mass by media fraction.

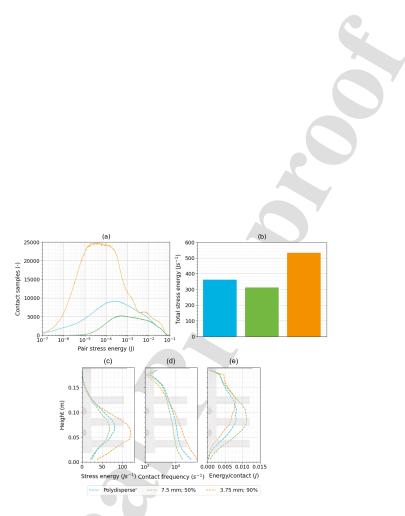


Figure 26: Energy and contact data comparing three scenarios: a polydisperse case by number, an equal count with a low size ratio, and a high proportion of small media with a high size ratio. (a) Logarithmic energy distribution per contact, (b) total stress energy per second, (c) energy distribution by height, (d) number of contacts across vessel height, and (e) energy per contact as a function of vessel height. All scenarios were run at 250 rpm.

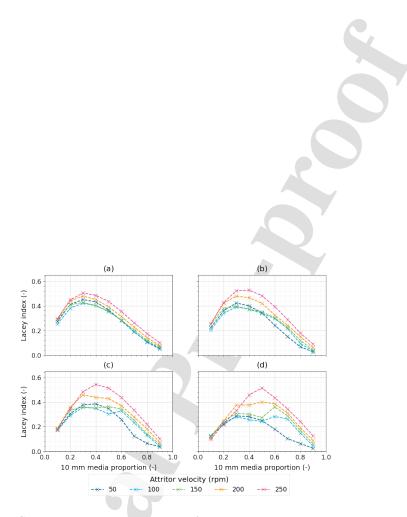


Figure 27: Calculated Lacey mixing indices for each simulation using $\varepsilon = 0.9$ and $\mu_s = 0.1$. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, and (d) 3.75 mm.



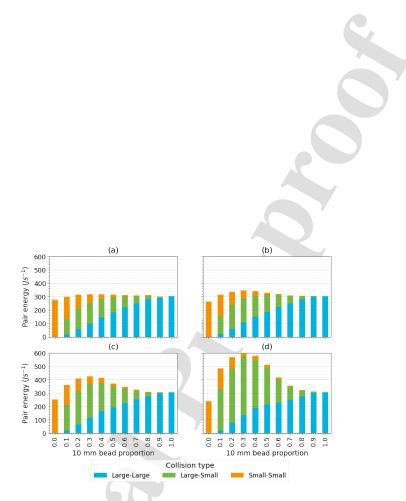


Figure 28: Overall collision energy spectra for different bead proportions and ratios at 250 rpm using $\varepsilon = 0.9$ and $\mu_s = 0.1$. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm and (d) 3.75

mm.

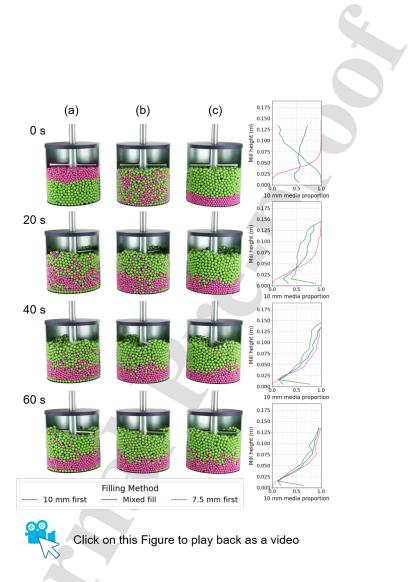


Figure 29: Progression of media motion over time, based on different starting fill conditions. All mills contain an equal number of 10 mm (green) and 7.5 mm (pink) media and were run at 50 rpm. (a) The large media are filled first, (b) the media are randomly filled, and (c) the small media are filled first. The plots on the right-hand side show the

fraction of $10\,\mathrm{mm}$ media by mill height.

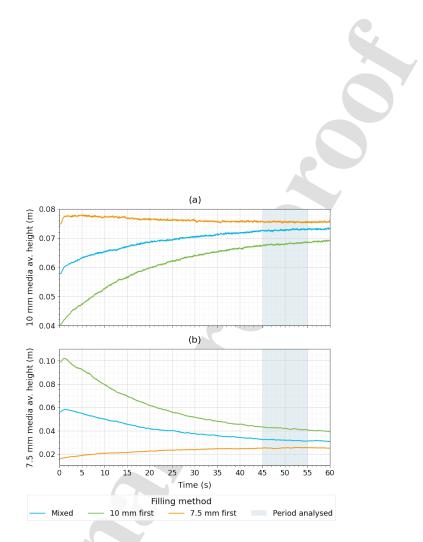


Figure 30: Progression of segregation over time after different starting fills using fraction centre of mass in the height axis. (a) 10 mm media, (b) 7.5 mm media. The shaded region between 45 and 55 seconds on each plot indicates the period analysed for all subsequent simulations.



Figure 31: The different regions analysed within the mill, with red shading denoting the areas not analysed. All measurements are in mm. (a) Full mill region, (b) the area containing the pins, (c) removal of a noticeable dead zone below the impeller.



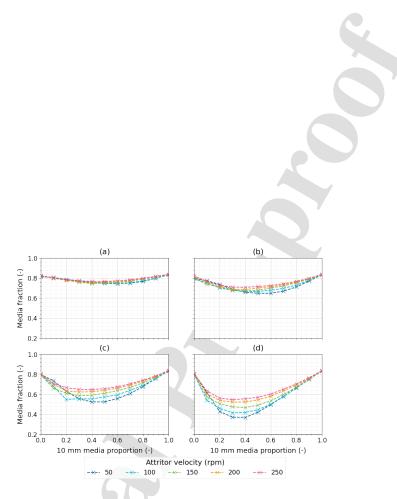


Figure 32: Fraction of all media by 10 mm bead proportion that are are above the bottom 20 mm of the milling space at different rotational velocities. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, (d) 3.75 mm.

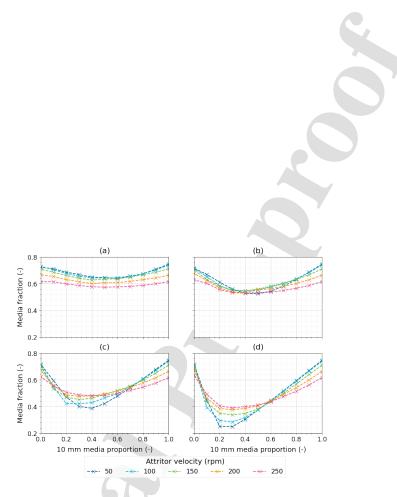


Figure 33: Fraction of all media that sit in-line with the attritor pins at different rotational velocities, plotted against the proportion of 10 mm media present. (a) 7.5 mm, (b) 6.25 mm, (c) 5.0 mm, (d) 3.75 mm.

Mechanism	Symbol	Equation
	J	1
Impact	E_{tn}	$E_{tn} = 0.5m(v_{ij}^n)^2$
	L_{ln}	
Torsion		
	E_{rn}	$E_{rn} = 0.2mR^2 (\boldsymbol{\omega}_{ij}^n)^2$
Shearing		$\mathbf{P} = (\mathbf{r} + \mathbf{r})^2$
	E_{ts}	$E_{ts} = 0.5m(v_{ij}^t)^2$
		Y
	E_{rs}	$E_{rs} = 0.2mR^2 (oldsymbol{\omega}_{ij}^t)^2$
	LTS	
	- 60	
	E_{tr}	$E_{tr} = 0.5m(\boldsymbol{v}_i + \boldsymbol{v}_j)^2$
Rolling		
		$E_{rr} = 0.2mR^2(\boldsymbol{\omega}_i + \boldsymbol{\omega}_j)^2$
	E_{rr}	

Table 1: The six Equations defining contact pair collision mechanisms [52].

Parameter	Valu
Attritor rotational velocity range (rpm)	50-25
Large media diameter (mm)	10.
Small media diameter range (mm)	2.5-7.
Total media mass (kg)	14.8
Fill level (%)	55.
Numerical proportion of large media (-)	0.0-1.
Media density $(kg m^{-3})$	785
Young's Modulus $(N m^{-2})$	2.1×10^{-10}
Poisson ratio, ν (-)	0.
Media-media restitution coefficient (-)	0.
Media-media sliding friction (-)	0.
Media-wall restitution coefficient (-)	0.
Media-wall sliding friction (-)	0.
Media rolling friction (-)	0.
Timestep (s)	10-
Recorded simulation time (s)	10.

Table 2: The fixed simulation parameters which were used for the investigation [36, 41]

Segregation in binary and polydisperse stirred media mills and its role on grinding effectiveness

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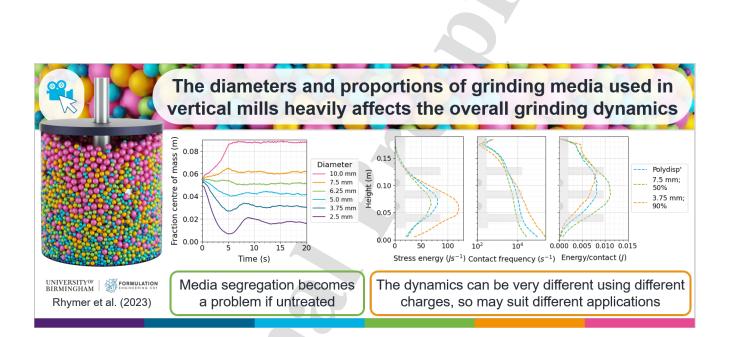
Paper Highlights

(Original)

- We investigate the effect of including multiple media sizes within a vertical mill.
- Changing size distribution can control the magnitude and frequency of media contacts.
- Power draw can be optimised, increasing the grinding sustainability.
- Natural segregation can be used to find ideal particle transport and flow patterns

(Revised)

- We investigated the effect of including multiple media sizes within a vertical mill.
- Having two sizes can increase stress energy in the mill by up to 55%.
- Power draw can also be reduced by up to 30% by including a second size.
- Segregation means high-impact and high-attrition contacts occur simultaneously.



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Credit Statement

Daniel Rhymer: Methodology, Model Development, Investigation, Data Curation,

Data Analysis, Writing - Review & Editing.

Andy Ingram: Supervision, Data Analysis, Reviewing & Editing.

Kieran Sadler: Study Conceptualisation, Supervision.

Kit Windows-Yule: Supervision, Data Analysis, Reviewing & Editing.

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Declaration of Competing Interest

The authors declare upon submission that there are no competing interests.