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## WAVE INDUCED SALINE INTRUSION IN SEA OUTFALLS

by

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### Summary

Experimental and numerical studies have shown that the influence of waves increases the tendency of saline intrusion in multi-riser sea outfalls. The flow field in the diffuser under such unsteady and inhomogeneous circumstances is in general very complex, but when sufficient wave energy is dissipated in the diffuser, a well-mixed flow regime occurs where numerical modelling can be applied. The numerical model seems to be able to calculate within a reasonable accuracy, the point where the intrusion begins.

### Introduction

The disposal of sewage to the sea through long sea outfalls has been under discussion for more than 25 years. The practice differs significantly from country to country. In U.K. is seen long sea outfalls which discharge untreated sewage several kilometers from the shoreline, whereas the Danish practice is almost the opposite with short outfalls combined with secondary or even better treatment. But despite these discrepancies in environmental aspects, it is a common problem to keep the outfalls in constant function.

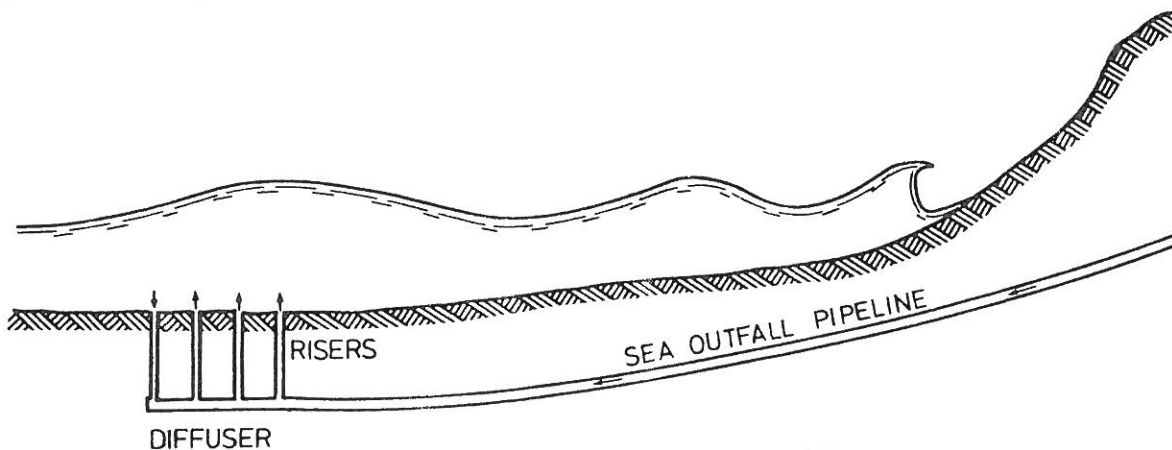


Fig. no. 1: Sketch of multi-riser outfall.

Saline intrusion has been recognized as a serious complication to the hydraulic performance of sea outfalls, Charlton 1985. The intruding sea water is supposed to increase siltation in the diffuser with partial blockage as the further consequence.

The aim of this work is to establish methods and criteria for the determination of the conditions where saline intrusion just starts. The methods presented here should be seen from this point of view more than an attempt to describe all aspects of the extremely complex unsteady and inhomogeneous flow in the purging itself.

Williamson, 1985, has shown that the densimetric Froude number for an individual diffuser port cannot be used as a criterion for saline exclusion in the steady state situation if earlier operation cycles have permitted saline influx, during shut-down for example. Instead it was suggested that purging could start only when the surplus weight of sea water in the outmost riser is exceeded by the head loss in the other risers. To expand this principle to the unsteady wave situation a numerical model is needed.

It has earlier, Larsen 1986, been reported that wave induced pressure field near the sea bottom can cause backflow through the diffuser ports. In this study the density effect has been added to the numerical model.

## Experiments

Physical model tests carried out at the University of Liverpool have been reported in detail earlier, Burrows, R. and Mort, R.B., 1988. The arrangement is shown in Fig. 2.

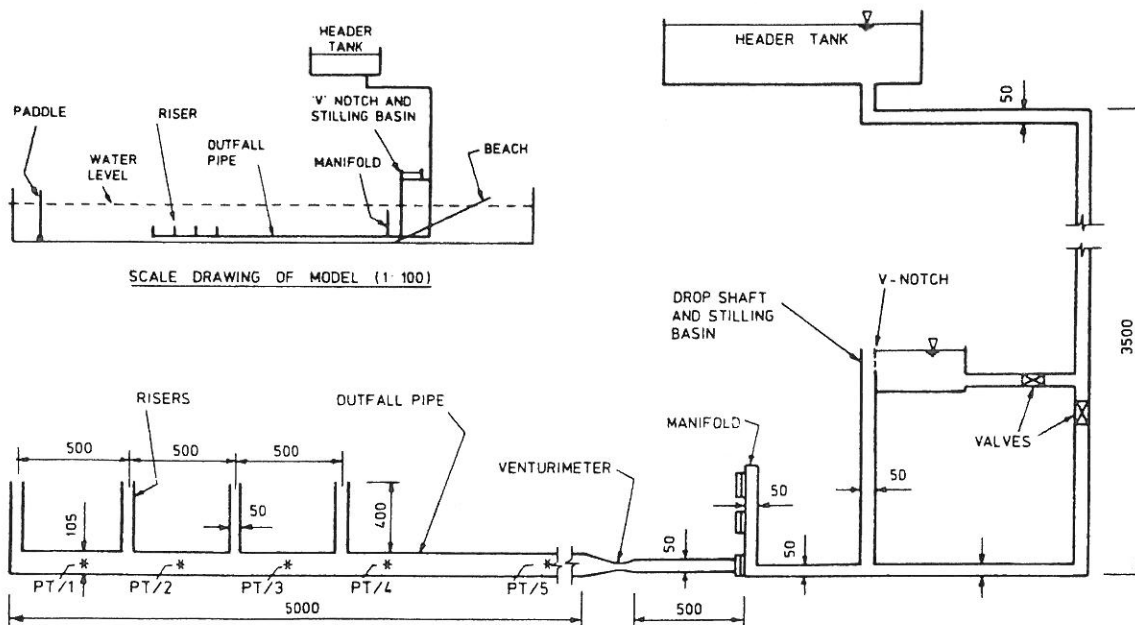


Fig. no. 2: General arrangement of experimental apparatus.

The experiments involve primarily the measurement of the riser velocities and an example is shown in Fig. 3.

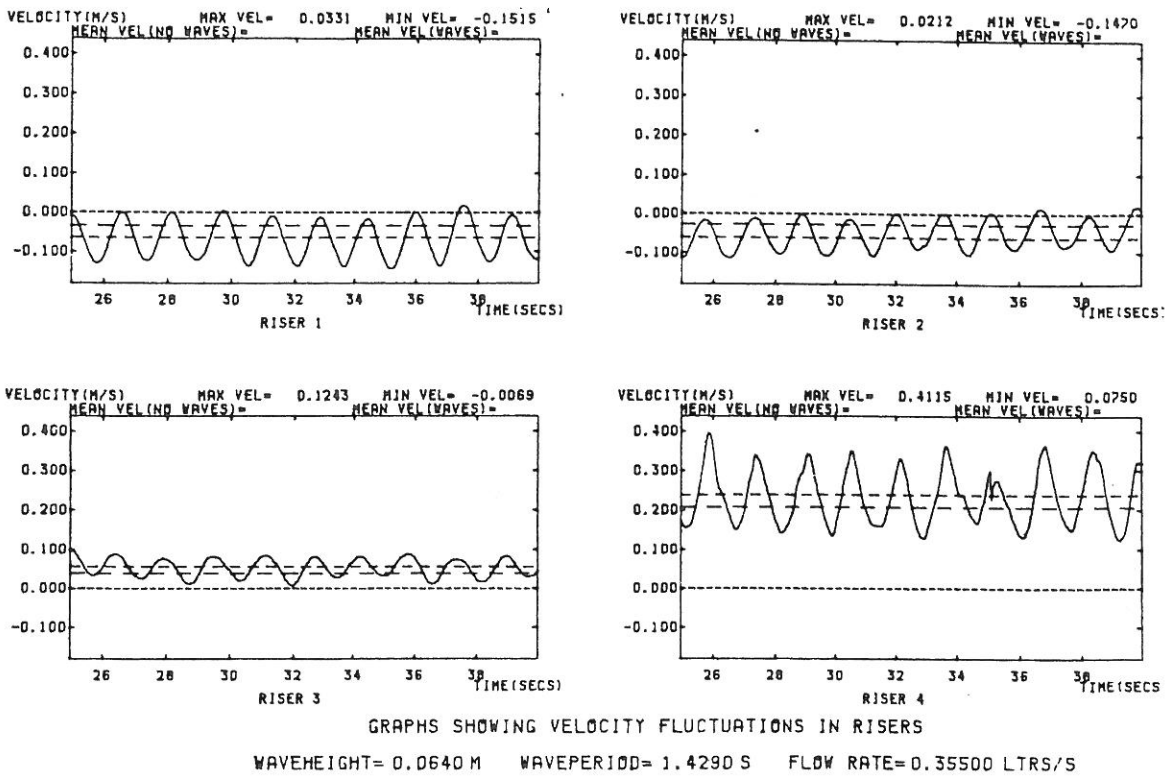


Fig. no. 3: Experimental velocity fluctuations in risers for discharge  $Q = 0.35 \cdot 10^{-3} m^3/sec$ .

A qualitative but important result of the model tests was the visual observations for the mixing in the diffuser based on dye injection to the fresh water flow.

In the steady state case without waves, it was typical that a saline wedge was formed in the horizontal diffuser pipe. Even the riser flow showed to be a two "layer" flow when the fresh water flow was low.

In the unsteady case, when the waves exceeded a certain height, this picture was totally changed and the flow became almost fully mixed with sea water in one part of the diffuser and fresh water in the other part, separated in a narrow mixing zone near one of the riser tees.

A quantitative experimental description of the transition between the above mentioned characteristic flow regimes is still under investigation and cannot be presented here. Further experiments with varying fresh water flow, wave height, and density difference will be done in 1989 and results will be published later.

### Numerical model

To describe the combined effect of waves and density in the well mixed flow regime a numerical

model was set up. This model couples a flow model based on the momentum and the continuity equation with a density difference model based on the transport/dispersion equation.

*Flow model*

The unsteady flow in the diffuser is a very complex phenomenon in general. From a physical analysis of force and energy balances it was concluded that

1. elastic effects of water and pipe could be neglected
2. acceleration terms were more important than friction and density differences
3. density differences did not affect the acceleration but only the pressure.

The momentum equation for each pipe section then becomes:

$$\text{Vertical riser} \quad : \quad \rho A \frac{\partial V}{\partial t} + \rho g A \frac{\partial h}{\partial x} + \tau P + \Delta \rho g A = 0$$

$$\text{Horizontal pipe} \quad : \quad \rho A \frac{\partial V}{\partial t} + \rho g A \frac{\partial h}{\partial x} + \tau P = 0$$

To include the head losses in tees and diffuser ports, the energy equation was applied at those points. The friction in the pipe sections was calculated from the Darcy-Weisbach equation. The gravity term in the riser equation gives the effect of density difference and the coupling to the salt model. Further details are seen in Wylie and Streeter, 1985.

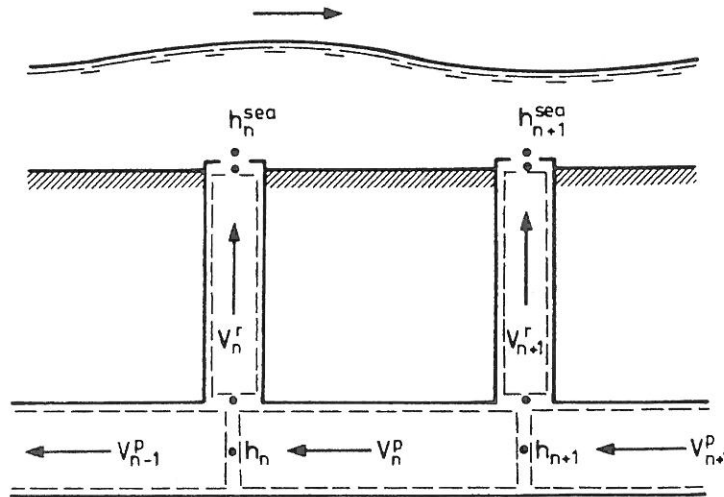


Fig. no. 4: Outline of rigid column flow model.

The unsteady boundary conditions just outside the diffuser port was the wave induced pressure fluctuations, which were calculated from linear wave theory.

The continuity in each tee was

$$Q_{n-1}^{pipe} - Q_n^{pipe} - Q_n^{riser} = 0$$

To solve the equations the momentum equations were substituted into the continuity equations. According to the above mentioned assumption no. 2 all friction and density effect could be taken on the old time step. The substitution then gives a linear equation for each tee in the form

$$A_n h_{n-1} + B_n h_n + C_n h_{n+1} = D_N$$

The simultaneous linear equations were then solved by the double sweep method to give the head on the new time step. Substituting back in the momentum equations then gave the velocities.

*Salt model*

The salt model was more precisely a mass balance model for the density  $\rho$  based on the transport/dispersion equation

$$\frac{\partial \rho}{\partial t} + V \frac{\partial \rho}{\partial x} = D \frac{\partial^2 \rho}{\partial x^2}$$

This model differs significantly from the flow model because it needs much shorter  $\Delta x$  to avoid numerical dispersion.

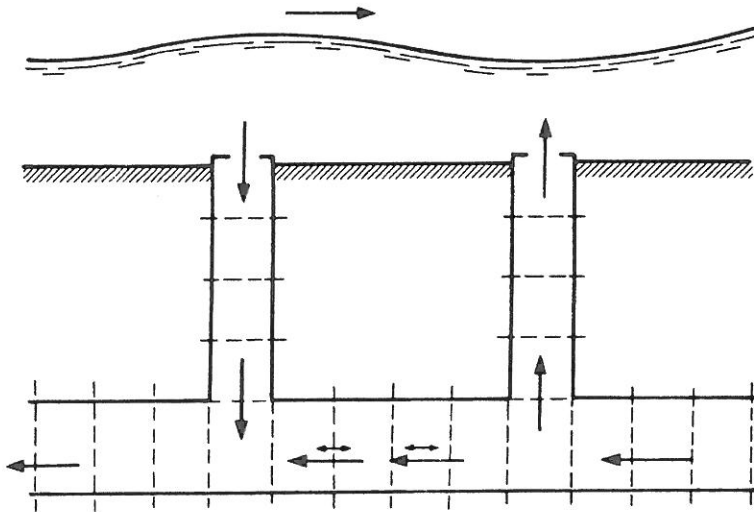


Fig. no. 5: Outline of transport/dispersion model

The finite difference solution of transport/dispersion is well-known and details will not be given here.

For numerical reasons a standard steady state dispersion coefficient for turbulent pipeflow was applied. As mentioned later the value of this parameter is unimportant.

*Results from numerical model*

All the results presented here refer to geometry, flow and wave situations equivalent to the model test described earlier. Only a few principal results will be shown.

Fig. no. 6 shows the velocities in the riser with no basic flow where the diffuser contains sea water only.

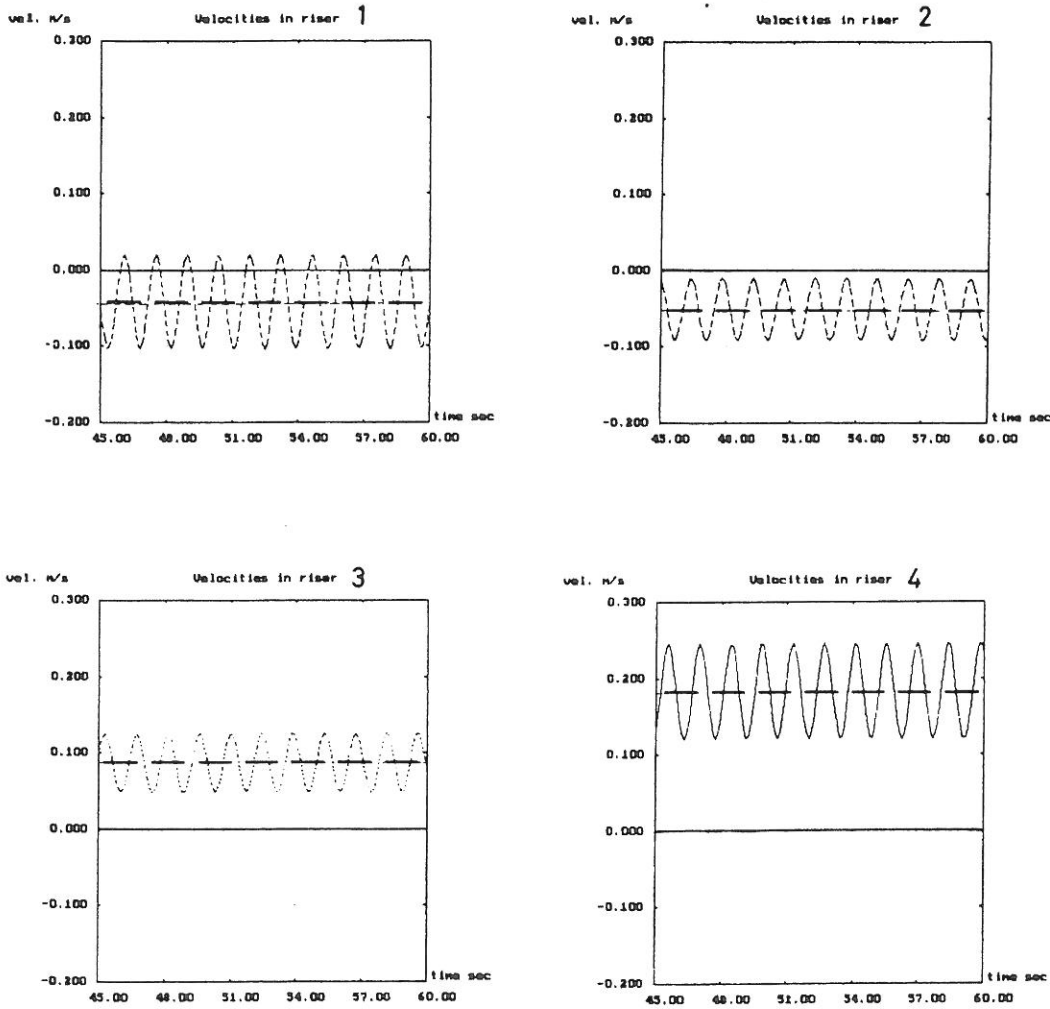


Fig. no. 6: Computed riser velocities equivalent to Fig. no. 3.

Fig. no. 7 shows the computed density distribution in the diffuser pipeline with a basic flow of  $Q_o = 0.00035 \text{ m}^3/\text{sec}$ , which shows a typical situation with saline intrusion in the diffuser. It is seen how the mixing takes in a narrow area between two tees in agreement with the observations in the model tests. The presence of this mixing area is the result of the oscillating flow, which indicates that most of the buoyancy energy is dissipated here.

Figure no. 7 shows the longitudinal density distribution along the diffuser pipe equivalent to Figure no. 6 and the steep gradient indicates the mixing area.

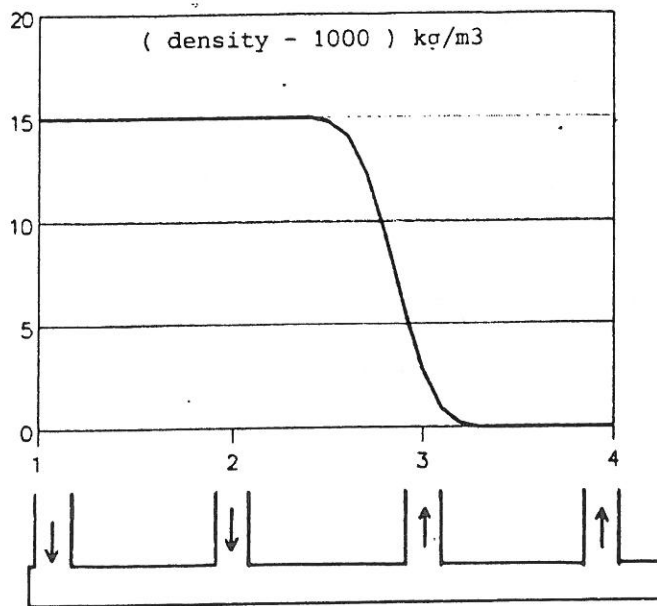


Fig. no. 7: Longitudinal density distribution in diffuser pipe equivalent to Fig. no. 6.

### Conclusions

1. Experiments and computations show that waves and density differences will induce unsteady internal flow in the diffuser and a net circulation will occur where sea water will purge into the outermost ports and out of the innermost.
2. The input of buoyant energy from the sewage discharge strongly increases the tendency for purging of sea water through the outermost ports. The numerical model seems to agree acceptably to the experiments.
3. Despite several simplifications, the numerical model seems to be a reasonable tool for the determination of the point where saline intrusion begins, even in the non-wave case.

### Acknowledgement

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