

## **Abstract**

Building energy and indoor air quality (IAQ) are of great importance to climate change and people's health and wellbeing. They also play a key role in mitigating the risk of transmissions of infectious diseases such as COVID-19. Building design with high performance in energy efficiency and IAQ improvement can save energy, reduce carbon emissions, and improve human health. High-performance building (HPB) design at the early design stage is critical to building's real performance during operation. Fast and reliable prediction of building performance is, therefore, required for HPB design during the early design iterations. A modular-based method to analyze building performance on energy efficiency, thermal comfort, IAQ, health impacts, and infection risks was developed, implemented, and demonstrated in this study. The modular approach groups the building technologies and systems to modules that can be analyzed at multi-scale building environments, from urban scale, to building, room, and personal scale. The proposed approach was implemented as a plugin on Rhino Grasshopper, a 3D architectural geometry modeling tool. The design and simulation platform was named Green Design Studio. Reduced-order physics-based models were used to simulate thermal, air, and mass transfer and storage in the buildings. Three cases were used as the study case to demonstrate the module-based approach and develop the simulation platform. Optimization algorithms were applied to optimize the design and settings of the building modules beyond the reference case. The case study shows that the optimal design of the small office determined by the developed platform can save up to 27.8% energy use while mitigating more than 99% infection risk compared to the reference case. It reveals that the optimization of green building design using the proposed approach has high potential of energy saving and IAQ improvement. In support of the application of the Green Design Studio platform, a database of green building technology modules for energy efficiency and IAQ

improvement was created. Two selected emerging IAQ strategies were studied using the proposed approach and the developed tool, including the in-duct needlepoint bipolar ionizer and the combination of displacement ventilation and partitions. The in-duct ionization system can provide an equivalent single pass removal efficiency (SPRE) of 3.8-13.6% on particle removal without significant ozone and volatile organic compounds (VOCs) removal and generation with minimal energy use. The combined application of displacement ventilation and desk partitions can also effectively mitigate potential virus transmission through coughing or talking. The abundant performance data from experiments and detailed simulations for the studied technologies will be used by the database of the green building technologies and systems. It will allow these two technologies to be applied through the Green Design Studio approach during the early-design stage for a high-performance building. This can potentially help to address IAQ issues, particularly the airborne transmission of respiratory diseases, while maintaining high energy efficiency.

GREEN DESIGN STUDIO: A MODULAR-BASED APPROACH FOR HIGH-  
PERFORMANCE BUILDING DESIGN

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## **1. Introduction**

### **1.1 Background and challenges**

Climate change already inflicts serious damage on the world. In recent years, we have seen more frequent and severe droughts, heatwaves, wildfires, storms, and flooding across the world (Cappucci and Kornfield 2022; Hoffman 2022; Office of Climate Change and Health Equity 2022; U.S. Census Bureau 2021). The ongoing extreme heatwaves in this summer are threatening millions of people on this planet (Cappucci and Kornfield 2022; NASA 2022; Office of Climate Change and Health Equity 2022). Experts have warned that these extreme weather events will worsen and become more frequent in the coming decades because of climate change (Hoffman 2022). In the meantime, the unprecedented COVID-19 pandemic brings more challenges to the world. Climate and health have become two major global crises, leading to devastating loss of life and economy. Many efforts have been made worldwide to address the global climate crisis and health issues, such as setting up carbon neutrality goals (European Parliament 2021; White House 2021b). But we are still facing many challenges to implement actions to achieve these goals.

Buildings play a significant role in energy consumption and climate change as the building sector is the largest energy consumer across other sectors, accounting for 35% of global energy use and 38% of carbon dioxide (CO<sub>2</sub>) emission (Figure 1-1) (United Nations Environment Programme 2020). As the carbon neutrality goal set up by the Intergovernmental Panel on Climate Change (IPCC), global net human-caused emissions of carbon dioxide would need to fall by about 45% from 2010 levels by 2030, reaching ‘net zero’ around 2050 (IPCC 2018). Therefore, the potential of carbon emission reduction in buildings is huge, but the challenges are also great. Building indoor environmental quality (IEQ) is closely associated with occupant health and comfort as people



spend nearly 90% of their time indoors (deCastro et al. 2007). Occupant exposure to indoor air pollutants, such as volatile organic compounds (VOCs), ozone, and particulate matters (PM), may cause short- or long-term adverse health effects, including increasing in respiratory-related morbidity, cardiovascular morbidity and premature mortality (Bell et al. 2004; Day et al. 2017; Gan et al. 2011; Tagiyeva and Sheikh 2014). The COVID-19 pandemic also highlighted the significance of indoor air quality (IAQ) as most outbreaks occurred in building indoor scenarios and airborne transmission plays a key role in virus spreading (H. Qian et al. 2020; Shen, Kong, Dong, et al. 2021b). Efforts to improve IAQ and comfort can improve people's health, well-being, and productivity. The increasing demand for healthier and more comfortable indoor environments is usually contradictory to building energy saving as increased energy use is typically required for space conditioning, ventilation, and air cleaning. Hence, it is essential for high-performance building design to achieve the trade-off between energy efficiency, and improved IAQ and thermal comfort.

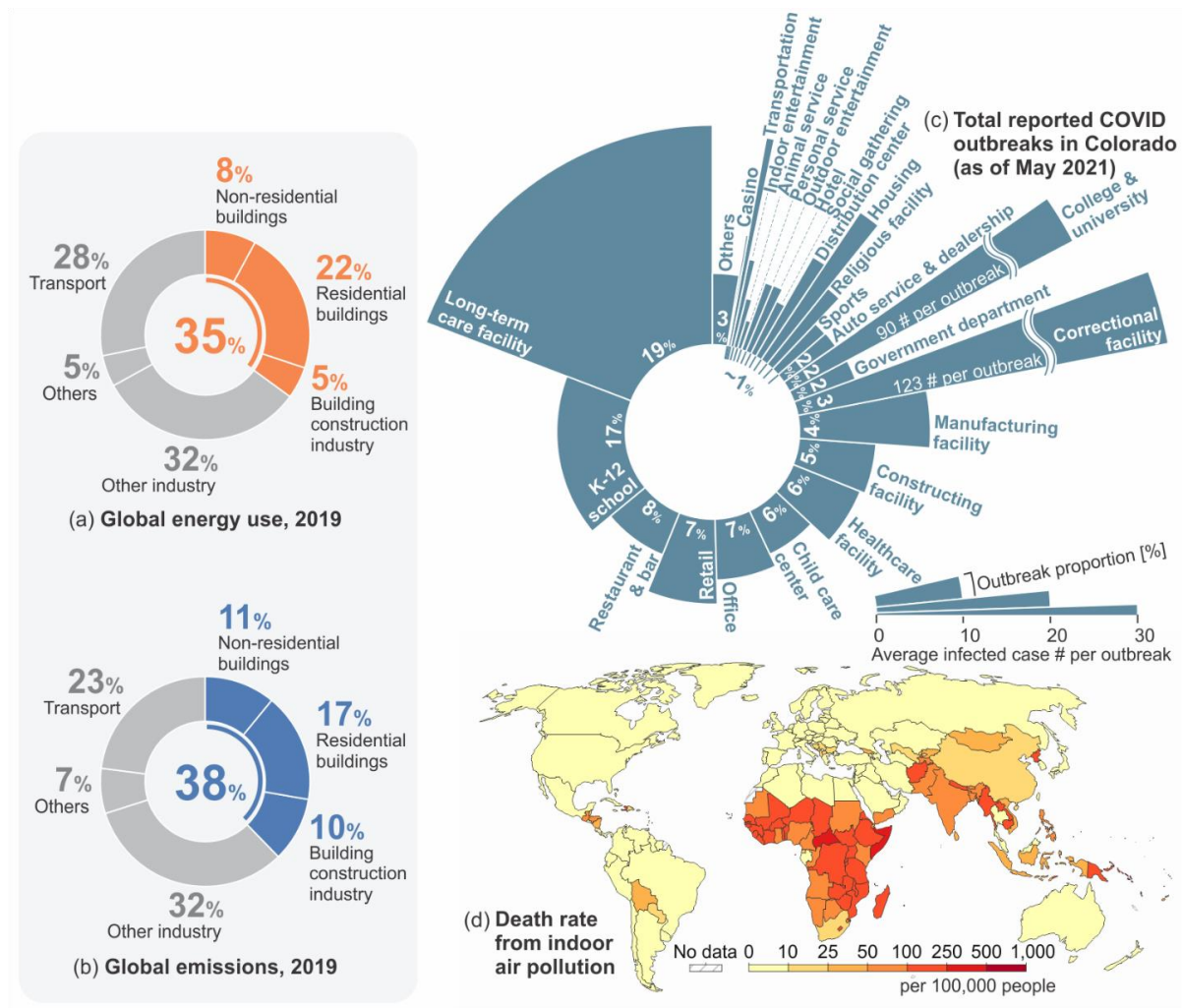


Figure 1-1. (a) Global energy consumption and (b) CO2 emission by buildings in 2019 (adapted from (United Nations Environment Programme 2020)), (c) COVID-19 outbreaks in different scenarios (data in Colorado, U.S. as of May 26th, 2021 (CDPHE 2020)), and (d) death rate from indoor air pollution (as of 2018) (Max Roser and Hannah Ritchie 2022).

The last two decades has seen rapid growth in high-performance building (HPB) design (also known as green building design or sustainable building design) with the emergence of many worldwide green building standards or certification/rating systems to minimize or optimize

consumption of natural resources and control pollution. It is estimated that there are approximately 600 HPB certification systems globally (Doan et al. 2017; Vierra 2018), including the most widely used ones such as LEED rating system (USGBC 2013), ASHRAE standard 189.1 (ASHRAE 2017c), BREEAM standard (BRE 2018), and WELL building standard (IWBI 2018). The LEED Rating System (USGBC 2013) is an internationally recognized green building certification system aimed at improving performance across all the metrics including energy savings, water efficiency, CO<sub>2</sub> emission reduction and IEQ improvement. It has become one of the most common green building certification systems in the world (USGBC 2022). Currently, more than 90 thousand building projects have been certified by the LEED system in more than 167 countries [13,19]. The BREEAM System (BRE 2018) is the first and another leading green certification system. It evaluates performance across the building lifecycle, from new construction to in-use and refurbishment. Nearly 600 thousand buildings have been certified by the BREEAM system in 90 countries (BRE 2019). The WELL Building System (IWBI 2018) is a more recent international certification system that aims to improve human health and wellbeing through the built environment. Over 9 thousand buildings have been certified by the WELL system globally (IWBI 2019). The U.S. has a large market for green buildings. Figure 1-2 shows top 10 countries and regions in terms of total LEED-certified floor area (square feet) in 2021 (U.S. Green Building Council 2022). The U.S. is the world's largest market for LEED, with more than 280 million square feet certified to LEED in 2021.

Although better performance is expected for the certified green buildings, the practical performance of green buildings may not always be consistent with the target rating level. A study analyzed the energy use from 100 LEED-certified buildings and found that around 30% of them

consumed more energy than their conventional counterparts, although on average, LEED-certified buildings saved 18-39% of energy use (Newsham et al. 2009). A recent study revealed that LEED-certified buildings consume only 10% less site energy than similar buildings that are not LEED-certified (Scofield and Doane 2018). Some green buildings may perform well on energy saving, water saving, carbon emission reduction, and operation cost reduction, but have poor performance on IAQ and thermal comfort. A more recent study reported average 25% less energy use, 11% less water use, 34% reduced CO<sub>2</sub> emission, and 19% lower operation cost for 22 LEED-certified green buildings compared to conventional buildings (Fowler et al. 2010). But the IAQ and thermal comfort in these green buildings were worse than conventional buildings (Fowler et al. 2010). Even the buildings that certified by the WELL standard, that aims to promote IAQ and occupant health, may exhibit higher measured concentrations for some VOCs and TVOC relative to non-WELL-certified buildings (Licina and Langer 2021).



Figure 1-2. Top 10 countries and regions in terms of total LEED-certified floor area (square feet) in 2021 (data from (U.S. Green Building Council 2022)).

Green technologies, strategies, facilities, or building components that perform better on energy efficiency, IAQ and comfort improvement, have been developed and widely implemented in buildings to achieve higher performance across multiple scales (from urban scale, to building and room scales, to personal microenvironment), e.g. green roof, superinsulation building envelope, low-emission building materials, and personalized ventilation system (Cao et al. 2016; Kong et al. 2015; Pétigny et al. 2021; Sadineni et al. 2011; Shen, Kong, Dong, et al. 2021b, 2021a; Tomson et al. 2021; J. J. Zhang et al. 2022). Different building systems can have completely different or even contradictory performance on energy efficiency and IAQ and thermal comfort. Even the same technology may perform differently under various conditions. For example, in-duct filters with higher efficiencies are usually suggested for removing indoor particles or mitigating the airborne transmission risk, but higher energy consumptions by the driving fan are usually required to

compensate the increased pressure drop through the filter (Risbeck et al. 2021; Shen, Kong, Dong, et al. 2021b, 2021a). Natural ventilation can enhance indoor ventilation, mitigate indoor air pollution and reduce the energy use by mechanical ventilation, but when the ambient air is not clean enough, it will likely introduce contaminants from the outdoor air like ozone and nitrogen oxides (NO<sub>x</sub>) into indoor air (Shen et al. 2020; Shen, Kong, Dong, et al. 2021a). To mitigate the airborne transmission risk of SARS-CoV-2, a higher fraction of outdoor air in the supply air of the heating, ventilation, and air conditioning (HVAC) system (e.g. 100% outdoor air) is usually recommended to improve the indoor ventilation, although it will consume more energy to condition the additional outdoor air (Shen et al. 2020; Shen, Kong, Dong, et al. 2021a). Enhanced ventilation can also break the equilibrium near the surfaces of building materials. It can lead to release of pollutants, which are primarily emitted by materials (intrinsic emissions) and also those that adsorb on surfaces and are absorbed in the building structures (Steinemann et al. 2017). Therefore, the selection and combination of various building technologies are significant to building performance. Inappropriate selection and use of some systems in green buildings may impair building performance. Hence, it is essential to address these issues at the design stage of green buildings.

Figure 1-3 illustrates a typical building design process. To aid decision-making of a HPB design, building simulation is widely used in the late design stages, but its application is still limited in the early stages in which design decisions have a major impact on final building performance and costs (Attia et al. 2012; Hygh et al. 2012; Konis et al. 2016; Larsson 2009). Predicting the consequences of early decisions is particularly difficult, but crucial, since adverse design decisions will reduce the remaining design space and make it more strenuous and expensive to meet high

performance goals (Østergård et al. 2016). Research shows that during early design phases, 20% of the design decisions taken subsequently influence around 80% of all design decisions (Bogenstätter 2010). The early integration of simulation programs faces several challenges, including time-consuming modeling, rapid change of the design, conflicting requirements, input uncertainties, and large design variability (Østergård et al. 2016). In addition, building design is a multi-collaborator discipline, where design decisions are influenced by architects, engineers, and clients (building owners) (Østergård et al. 2016). Several studies have shown that current design and decision support tools are inadequate, user hostile and too incomplete to be used by architects to support and inform the HPB design, particularly during the early design phase (Attia et al. 2009, 2012; Riether and Butler 2008; Stine 2022). Per the building performance simulation (BPS) tools originally listed by the U.S. DOE, out of the total 392 tools listed on the DOE website in 2011, less than 40 tools are targeting architects during the early design phases (Attia et al. 2012; BESTD 2022). To address these challenges for early-phase HPB design, two approaches can be implemented. One is developing a program that can provide real-time or near-real-time and reliable feedback of building performance simulation results to meet the rapid changes of the design. Another possible approach is to establish a comprehensive database of various building features/technologies/systems with pre-defined parameters and pre-calculated/pre-measured performance metrics embedded. During the early design stage, the pre-calculated/pre-measured information can provide direct guidance on the selection and configurations of different building features in HPB designs.

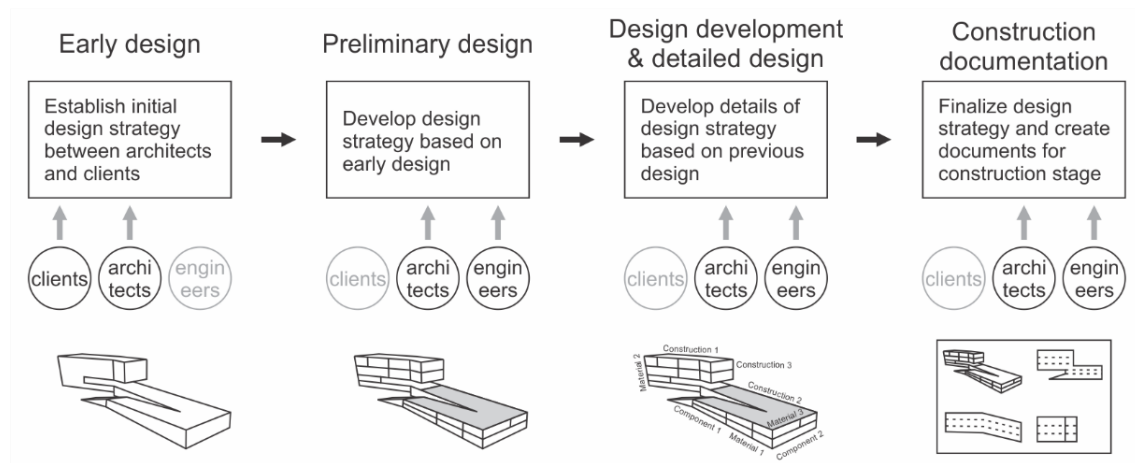


Figure 1-3. Schematic of a typical building design process.

Both approaches require quantitative analyses on various building performance metrics. Therefore, quantitative and comprehensive analyses on the performance and cost of different green building technologies in realistic scenarios are important for building design to achieve the performance goal, which can provide useful supports and reference to designers for making decisions on selecting green technologies and systems in the design. Building performance simulation can support decision making in early design of high-performance building as they can provide quantitative analysis to designers (Østergård et al. 2016). Multiple simulation tools have been developed and applied to evaluate various building performance, including EnergyPlus (DOE 2017), eQuest (JJH 2018) (based on DOE-2 (Hirsch 2016)), ESP-r (ESRU 2018), and TRNSYS (TRNSYS 2019) for building energy efficiency modelling, Radiance (LBNL 2017) for lighting simulation, CONTAM (NIST 2018) and CHAMPS-Multizone (Feng et al. 2012) for multizone IAQ and ventilation analysis, and computational fluid dynamics (CFD) tools (e.g. ANSYS Fluent (ANSYS 2021) and OpenFOAM (OpenFOAM Ltd 2023)) for room air distribution simulation. These tools are usually developed based on physical equations of thermal and mass (air and



contaminants) models in building physics, which likely include some complicated equations and advanced algorithms for solving the coupling of different variants in the equations. These equations and algorithms can provide more accurate and reliable simulation results, but also require very detailed and complicated inputs such as detailed building envelope properties or HVAC system parameters. However, some of these detailed input parameters may not be available yet at the early stage of a building design. Failure to provide accurate input can result in poor prediction performance. In the meanwhile, it may consume considerable processing and simulation time for models with large and complex inputs, which is unfavorable for early-phase building design as there are always rapid changes to the building design at this phase, so real-time or near-real-time feedback of building performance estimation is required (Shen et al. 2020). Besides, most of these building physics modeling tools are specific to a certain task, e.g. EnergyPlus on energy simulation and CONTAM on IAQ simulation. It is not quite convenient and straightforward to run multidisciplinary modeling with these tools. External platforms or tools (e.g. Building Controls Virtual Test Bed (BCVTB) (LBNL 2016)) are usually needed to run the coupling simulations (or co-simulations) with these tools. Therefore, a tool that can perform multidisciplinary building performance simulations for early design stage that requires minimal inputs and fast response is necessary.

A database of green building technologies on various performance metrics (including energy efficiency, IAQ, comfort, and cost) will be beneficial for HPB design but is still lacking. Many studies have analyzed the performance for certain technologies, but a systematic analysis for them under various conditions is scarce. A comprehensive review of the performance analyses for different green building technologies is thus required.

To optimize the application of various building technologies/systems in HPB design, performance simulation tools can also collaborate with some optimization algorithms to determine their optimal configurations (Kheiri 2018; Nguyen et al. 2014). Multi-objective optimization (MOO) is widely used to optimize the building features to achieve the trade-off between different performance criteria, e.g. energy efficiency, IAQ, thermal comfort, and cost (Chegari et al. 2021; Diakaki et al. 2008; Ghaderian and Veysi 2021; W. Li et al. 2021; Hongbin Liu et al. 2013; Shaikh et al. 2018; N. Wang et al. 2014; X. Wei et al. 2015; B. Wu et al. 2021). However, building performance optimization (BPO) usually requires numerous iterations to meet certain criteria, which will consume considerable time and is less likely to give a fast response for early design. Hence, fast and reliable performance simulation and optimization models are required for early-phase HPB design.

Therefore, the main challenges for early-phase HPB design include:

- a) Current green building design standards and rating systems cannot provide quantitative and comprehensive support for high-performance building design, which may result in the actual building performance not achieving the target green level.
- b) Most building design and performance analysis platforms require complicated inputs and considerable processing time and cannot provide fast responses for building performance simulation and optimization at early design stage and in operation.

- c) A database that contains comprehensive analysis for green building technologies and systems, can provide support to designers for early-stage high-performance building design but is not available.

Hence, a building design and analysis platform that can provide fast and reliable responses for building performance simulation and optimization, is required. A green building technology/system database with comprehensive and systematic analysis on building performance also needs to be established in support of the design and analysis platform.

## **1.2 Research gaps**

Based on the review on existing building performance databases, the main research gaps of the existing building performance databases include: (1) Existing whole building performance databases usually only provide very limited building characteristics information, typically including floor area, built year, and floor number. Building components or systems with design features were usually not provided; (2) Most databases focused on one performance aspect, either energy or IAQ. Datasets with metrics of multiple performance aspects (energy and IAQ) are still scarce; (3) Databases of building components or systems usually do not provide performance metrics like energy use or carbon emissions for each component/system. Therefore, a database of building technologies, components, or systems with design features and performance metrics is required for the development of a BPS program for early-phase design (adopting design features as inputs for physics-based modellings and developing data-driven models to predict performance metrics).

### **1.3 Goals and objectives**

The research goal of this dissertation is to develop an approach for designing buildings with improved performance at the early design phase through 1) a framework for high-performance building design, 2) advanced algorithms for building performance simulation that can provide fast and reliable response, 3) a database of green building technologies and systems from real and validated practices, and 4) an optimization model that can coordinate different green features to achieve an optimal performance. The performance metrics that the proposed approach aims to improve include energy efficiency, IAQ, thermal comfort, and airborne infection risk for respiratory diseases, with the consideration of costs. The key scientific contributions will stem from a framework used for representing building components/systems and quantifying building performance metrics, a high-performance building design and optimization program with advanced simulation and optimization algorithms, and the comprehensive green building technology/system datasets. The proposed framework regularizes the parameters of various types of building components/systems and simplifies the inputs for performance analyses at the early building design phase. It also proposes a quantitative assessment approach for comprehensively analyzing the performance of buildings and building components/systems. It works for different scales of building systems, from community, site, building, floor, room, to personal microenvironment. The program with advanced simulation models can provide fast and reliable response of performance metrics, which is required for early-phase building design. The database of green building technologies and systems can provide supporting data for building design and be used to improve the data-driven simulation models. The performance optimization models can be used to find the design with optimal configurations of building technologies and systems.

Therefore, the proposed approach and platform can be used as a decision tool at the early design stage for the use of various green building technologies and systems in high-performance building design. Since the platform considers the infection risk of respiratory diseases through airborne transmission, it can also be used to estimate the infection risk in different scenarios, and evaluate the effectiveness of various mitigation strategies, which can provide helpful supports for selecting favorable control strategies in addressing respiratory diseases such as COVID-19. The platform is also well-suited for controlling the operation of different green building technologies and systems as control parameters and feedback metrics are defined and readily to be implemented in operation with the application of Internet of Things (IOTs). The hypothesis of the proposed research is that innovative and optimized high-performance building design has a great potential for significant energy saving, and occupant health and well-being improvement, and an approach with fast and reliable performance prediction models can significantly improve the building design and operation.

#### **1.4 Research scope and roadmap**

The research aims to develop an approach for designing green buildings at the early design phase.

The present work is limited to the following research scope:

- (1) Developing an approach that can be used to design green buildings and predict performance metrics.

- (2) Developing an approach to establishing a database of green building technologies or components with design features and performance metrics that can be further used by data-driven models.
- (3) Focusing on the performance modelling for early-phase building designs that require simplified modelling inputs and real-time or near-real-time response.
- (4) Demonstrating building performance modelling and green building design with some representative study cases.

To simplify the inputs for building performance modelling and establish the database of green building technologies, building technologies, components, or systems can be defined as building modules with both design features and performance metrics. Therefore, the modular approach is applied to modularize and regularize building technologies, components, and systems in this study. The research will be performed through 1) a framework for high-performance building design, 2) advanced algorithms for building performance simulation that can provide fast and reliable response, 3) a database of green building technologies and systems from real and validated practices, and 4) an optimization model that can coordinate different green features to achieve an optimal performance. Figure 1-4 shows the roadmap of accomplishing this research. The key scientific contributions will stem from the comprehensive green building technology/system datasets, a modular framework used for representing building components and systems, and a module-based high-performance building design and optimization platform with advanced simulation and optimization models. The proposed modular framework regularizes the parameters

of various types of building components and systems and simplifies the inputs for performance analyses at the early building design phase. It works for different scales of building systems, from community, site, building, floor, room, to personal microenvironment. The module-based platform with advanced simulation models can provide (near) real-time and reliable response of performance metrics prediction, which is required for early-phase building design. It also enables members of a multidisciplinary design team (architects, engineers, and clients) to provide inputs from different perspectives concurrently and therefore develop building designs and analyses easily and quickly through the same platform. The database of green building technologies and systems can provide supporting data for building design and be used to improve the data-driven simulation models. The performance optimization models can be used to find the design with optimal building technologies and systems. Hence, the proposed platform can be used as a decision tool at the early design stage for the use of various green building technologies and systems in high-performance building design. As the platform considers the infection risk of respiratory diseases through airborne transmission, it can also be used to estimate the infection risk in different scenarios, and evaluate the effectiveness of various mitigation strategies, which can provide helpful supports for selecting favorable control strategies, particularly during the COVID-19 pandemic. The platform is also well-suited for controlling the operation of different green building technologies and systems as control parameters and feedback metrics are defined in modules, which is readily to be implemented in operation with the application of IOTs. The hypothesis of the proposed research is that innovative and optimized high-performance building design has a great potential for significant energy saving, and occupant health and well-being improvement, and a modular based approach with fast and reliable performance prediction models can significantly improve the building design and operation.

The proposed research was accomplished through the following subtasks:

**(1) Module-based building design and analysis platform (R1):** A module-based design and analysis platform called Green Design Studio (GDS) is proposed. The proposed modular approach is applied to modularize and regularize building components and systems with building parameters and performance attributes embedded. A database containing a bunch of building modules is established and used by the platform to enhance the simulation algorithms and provide supporting data to building design. Reduced-order physics-based models and data-driven models are established and used to provide fast and reliable response on performance simulation. The database of green modules is used to improve the data-driven models. Expected outcomes include: (1) a modular framework and procedure to modularize and regularize building components and systems; (2) a module-based platform that requires simplified inputs and provides fast and reliable performance simulation response; and (3) advanced building performance simulation models, including reduced-order models and data-driven models.

**(2) Database of green building technologies and systems (R2):** A database of building modules with green features from real building practices is developed. The green building technologies and systems implemented in existing buildings will be modularized using the proposed modular framework to collect their configuration parameters. The performance attributes of each module are analyzed by detailed physics-based simulation models or verified and reliable reduced-order physics-based models. The performance attributes can also be originated from the measured data



when they are available in the studied building. The measured data can also be used for improving the reduced-order models cooperating with certain optimization algorithms. The established database will be used to improve the data-driven models for performance simulations. It can also provide information of how each module work in practice in different conditions and scenarios through certain statistical analyses, which can be used to provide supporting reference to designers for selecting green modules that corporate better with the building design. The expected outcome is the database of green building modules.

### **(3) Building performance optimization models coordinating various building modules (R3):**

Simulation-based building performance optimization models are developed to integrate and coordinate building modules to find out the design with optimal performance. Multi-objective optimization models will be developed to optimize the design and operation of building modules to achieve the tradeoff between energy efficiency, IAQ, thermal comfort, airborne infection risk mitigation and cost with certain constraints. The expected outcome is the building performance optimization models.

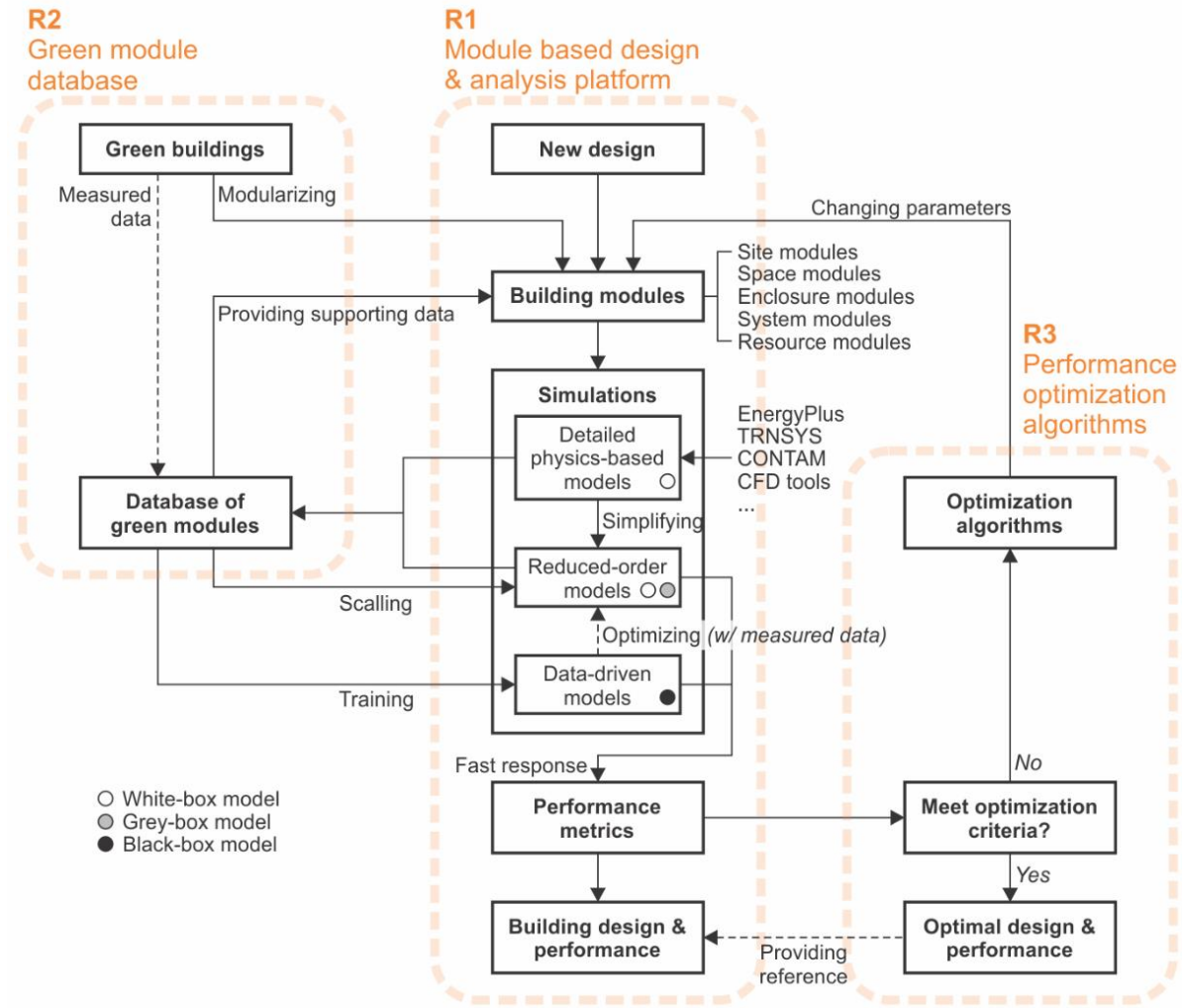


Figure 1-4. Roadmap of the proposed research.

## 2. Platform development for energy and IAQ analyses

### 2.1 Review of state-of-the-art research

#### 2.1.1 Performance analysis tools/models in building designs

There have been numerous programs developed for simulating various building performance metrics including energy efficiency, IAQ, thermal comfort, and infection risk. Many of these BPS tools have been widely applied in building designs. Simulation models in BPS tools can be roughly

categorized into three types, i.e. physical/engineering models, statistical methods, and hybrid approaches (Foucquier et al. 2013; Fumo 2014).

#### **2.1.1.1 Physics-based models**

Physics-based models (“white box”) usually can be implemented through different approaches, including the CFD approach for detailed thermal/air/mass distribution modelling in a space (treating the space as a 3-D problem), the zonal method that divides a space into multiple zones (treating the space as a 2-D problem), and the nodal method that assumes uniform state variables in the space (treating the space as a 1-D problem) (Foucquier et al. 2013). The CFD and zonal methods can provide spatial and time distribution of local state variables (e.g., air temperature or pollutant concentration) in the target space, while the nodal approach can just present the uniform values for the entire space. However, considering that a building usually consists of multiple rooms, CFD and zonal models for the whole building simulation usually require huge computation time with high complexity of the model implementation (especially CFD models). Therefore, the nodal method is widely applied in many building energy and IAQ simulation tools. The application of nodal models in buildings with multiple spaces can provide multizone simulations for energy and IAQ analyses. Most available building energy and IAQ programs are based on the nodal/multizone analyses, most typically EnergyPlus and CONTAM. For multizone models, the state variables can be estimated by a series of governing equations for calculating the thermal or mass balance in the space. Some simulation programs apply more comprehensive models to calculate the detailed thermal or mass distributions, e.g., the detailed construction layers in EnergyPlus, and the detailed HVAC systems in EnergyPlus, ESP-r, eQuest, and TRNSYS. The complexity of these detailed models will determine the accuracy and simulation time of applying these programs. Detailed

physics-based BPS tools like EnergyPlus and CONTAM have been widely applied in various research to evaluate the performance of different design features or novel technologies/systems (Al-Janabi and Kavgic 2019; Attia et al. 2012; Ng et al. 2013, 2021; Persily et al. 2010; S. Shi et al. 2015; Soares et al. 2014; Zhu et al. 2013).

These BPS programs can provide reliable and accurate predictions on different aspects of building performance, but the integration/coupling of various BPS programs is not very convenient or straightforward. External platforms or tools are usually required to run the co-simulations with these tools. BCVTB is a software environment that allows users to couple different simulation programs for co-simulation (e.g. EnergyPlus, Radiance, and TRNSYS), and to couple simulation programs with actual hardware (e.g. the BACnet system of a building) (LBNL 2016). It allows expert users of building simulation to expand the capabilities of individual simulation tools by linking them to other tools. It has been widely used by researchers and engineers to run co-simulations of building physics and control systems. Other tools that can export simulation programs as Functional Mock-up Units (FMU) using the Functional Mock-up Interface (FMI) standard (an interface to exchange dynamic models) also have the potential to perform co-simulations (FMI-standard 2022). For example, the EnergyPlusToFMU package is a package in Python that can export EnergyPlus as FMU for dynamic co-simulation or control (LBNL 2021). Simulations between EnergyPlus and CONTAM can be coupled through the CONTAM 3D Exporter that supports FMU (NIST 2022). Another significant approach for building physics co-simulation is programming in Modelica using the Building Library, which is an open-source library with dynamic simulation models for building and district energy and control systems (LBNL 2022). It is a more open platform/environment for building physics modeling, but usually

requires expert backgrounds on building physics and control systems. The co-simulation of building energy and IAQ performance is a widely studied topic since they are the major tasks of building performance. The co-simulation of energy and IAQ is usually performed through the coupling between EnergyPlus and airflow simulation programs like CONTAM for multizone airflow estimation (Dols et al. 2021; Justo Alonso et al. 2022; Ng et al. 2021) and CFD tools (e.g. ANSYS Fluent and OpenFOAM) for detailed airflow patterns in certain spaces (W. Guo et al. 2022; Xiong and Chen 2021). In general, the co-simulation of building energy, IAQ and other performance tasks is usually complicated and requires expert knowledge and experience in building physics modelling or programming.

Some simplified physics-based models have also been applied to simulate energy and IAQ performance with certain assumptions and simplifications (Rahman 2019; Xu and Wang 2007). Reduced-order resistance-capacitance (RC) thermal or air network models have been applied to simulate the thermal, air and mass transfer and balance in buildings (Braun and Chaturvedi 2002; Hao 2019; D. W. Kim et al. 2013; Z. Li et al. 2021; Mirakhorli and Dong 2016; O'Neill et al. 2010; O'Neill and Narayanan 2014; Ogunsola et al. 2014; Rahman 2019; S. Wang and Xu 2006; Xu and Wang 2007; K. Zheng et al. 2016). The RC network (including 3R2C and 2R2C models) is used to derive a set of first order differential equations representing the building thermal, air, and mass transfer and balance. Based on the RC thermal network, the energy balance equation for building envelopes has been applied to get the state-space model. In general, the resistances in the RC network cover outside convection resistances, wall conduction resistances, inside convection resistances, and window thermal resistance. The thermal mass effects are represented by the capacitances in the RC model. The network also includes the external and internal heat gains/losses

that apply on external or internal thermal nodes. For the airflow network based on the RC approach, the room air infiltration, natural ventilation, mechanical ventilation, contaminant sources and removals are considered. Research has shown that the RC network can describe the building heat transfer and a model with higher quality can give a better estimation accuracy (Hao 2019; D. W. Kim et al. 2013). The RC thermal/airflow network models have also been applied to integrate advanced control strategies such as model predictive control (MPC) in buildings (Boodi et al. 2019; Mirakhorli and Dong 2016) or work with other statistical methods such as genetic algorithm (GA) to establish hybrid models based on measurements or other datasets for better predicting building performance (Hao 2019; Xu and Wang 2007). In general, the reduced-order RC network models can provide fast and relatively accurate estimations on building performance with simplified inputs. Therefore, it is probably more suitable to apply reduced-order models in the early design phase as many physical information of buildings or systems required by detailed physics-based models/tools is unavailable at early design phase, which may result in poor prediction performance if failed to provide accurate inputs. Besides, the detailed physics-based models/tools usually consume more processing and simulation time, which is unfavorable for early-phase building design as there are always rapid changes to the building design at this phase, so real-time or near-real-time feedback of building performance estimation is required.

#### **2.1.1.2 Data-driven models**

Statistical methods using machine learning (ML) models (“black box”) have been used to estimate building performance more and more frequently in recent years. Compared to the physics-based models, the statistical models usually do not require physical information about the building or systems but estimate performance metrics based on existing datasets from measurements or

validated simulations (i.e., data-driven modelling). Based on the historical data, with suitable model and algorithms, statistical methods could “learn” the non-linear relationship between the independent variables and target variables (Zhijian Liu et al. 2019). This approach typically includes regression models such as multiple linear regression (MLR) and ordinary least squares regression (OLS), models based on decision trees, artificial neural network (ANN), and classifier models like support vector machine (SVM) (Foucquier et al. 2013; Seyedzadeh et al. 2018; W. Wei et al. 2019). Numerous data-driven studies have been conducted to predict and optimize building energy and IAQ performance (Foucquier et al. 2013; Seyedzadeh et al. 2018; W. Wei et al. 2019). According to a literature review on data-driven building energy consumption predictions published in 2018 (Amasyali and El-Gohary 2018), most studies, i.e., 47% and 25% of the studies, utilized ANN and SVM, respectively, to train their models. Only 4% of the studies utilized decision trees and 24% of the studies utilized other statistical algorithms such as MLR and OLS. These algorithms are also widely used in IAQ-related predictions (W. Wei et al. 2019). The accuracy of energy and IAQ predictions using ML methods usually depend on the suitability of training datasets and the selection of input variables/features, as well as the algorithms used to train the model. For energy simulations using data-driven models, building energy use/demand and heating/cooling loads are usually the target variables as outputs. Input variables of training datasets usually include meteorological and climate data (e.g. outdoor temperature, humidity, solar radiation, pressure, wind speed and direction), time information that associated with meteorological and occupant information (e.g. predicted season and date), building characteristics (e.g. stories, room features, construction year, orientation, geometry information such as area, height, and shape, window-to-wall ratio, glazing features, envelope characteristics, and shading characteristics, etc.), occupant information (e.g. number, density, and behavior), and active system

characteristics (e.g. HVAC, lighting, and ventilation system features and operations). Some studies also used historical data of the target/studied variables (i.e. energy use or heating/cooling loads) to predict the variables at the “current” moment (Amasyali and El-Gohary 2018; Seyedzadeh et al. 2018). For IAQ-related predictions, the output variables (or target variables) that were studied mainly include indoor concentrations of various pollutants (e.g. PM<sub>2.5</sub>, PM<sub>10</sub>, CO, CO<sub>2</sub>, NO<sub>2</sub>, airborne bacteria, total VOCs) or some IAQ indices consisting of multiple IAQ parameters (e.g. air quality index (AQI)), of which the PM concentration in indoor air is the most frequently studied parameter (W. Wei et al. 2019). Input variables of data-driven models for IAQ predictions are usually like the input features of models for energy predictions, including meteorological and climate parameters, time information, building characteristics, occupant information, and active system characteristics. Outdoor pollutant concentrations of the studied or other pollutant(s) are also commonly used as the input variables to predict the indoor concentrations of target pollutant(s). Besides, previous/historical indoor concentrations of the studied pollutant(s) can also be used to the indoor pollutant concentrations at the “current” moment (W. Wei et al. 2019). Research revealed that occupant behavior plays a crucially important role in building energy consumption and IAQ performance due to its high variability (Amasyali and El-Gohary 2021; W. Wei et al. 2019). A study presented that occupant behavior can make a difference up to over 7 times in energy consumption (Amasyali and El-Gohary 2021).

In general, data-driven models usually use more general parameters of building, meteorological, system, and occupant characteristics or historical data as inputs in their models, compared to the detailed and comprehensive parameters required by physics-based models. Therefore, the quality of training datasets can greatly affect the prediction accuracy of the data-driven models. Training



datasets are usually obtained through real measurements from sensors or simulated results from validated cases (e.g., validated CFD or EnergyPlus cases). To evaluate the performance of different building design features, datasets with building and system characteristics are required. However, many available public databases for building energy/IAQ predictions do not provide datasets with these features. In addition, to provide more accurate predictions, a large scale of datasets is usually required, which needs great efforts to generate and collect this number of datasets with studied features if no available databases existed.

Many studies have compared the performance of different data-driven algorithms on predicting energy and IAQ performance, which can vary greatly depending on the model nature (e.g., SVM vs ANN) and model features like hidden layer number for ANN models. Thus, the accuracy and speed of a specific data-driven model will vary from case to case. For example, many studies review by Amasyali and El-Gohary (Amasyali and El-Gohary 2018) revealed that SVM models perform higher accuracy than ANN models but some other studies observed the opposite. Many models can achieve high accuracy with the coefficient of variation (CV) below 5%, while others exhibited lower accuracy with very high CV (even greater than 100% for some cases) (Amasyali and El-Gohary 2018). According to the ASHRAE Guideline 14 (ASHRAE 2014), an hourly prediction model is considered calibrated if its hourly CV values fall below 30%. It indicates that the performance of using a data-driven model for prediction can vary significantly. For the same type of models, more layers or features usually result in higher accuracy, but also require greater training time. But prediction time of utilizing a pre-trained data-driven model is usually very fast, normally faster than detailed physics-based models. The choice of the algorithm to use, therefore,

depends on the application requirements (e.g., needed accuracy) and constraints (e.g., data availability) (Amasyali and El-Gohary 2021).

Even though data-driven models can provide fast and relatively reliable estimation of energy and IAQ performance, there may still be some limitations with the use of data-driven models. The models may not perform well outside of their training range, therefore, may not be reliable to be used beyond the training range (or beyond the studied building cases). For example, a model that was trained by learning from a limited dataset (e.g., data collected from a small set of buildings) may not perform well outside of the training data (e.g., different types of buildings in terms of physical properties, operation strategies, weather conditions, occupant behavior, etc.) (Amasyali and El-Gohary 2018). The dataset used for training must, thus, be representative of the range of application and contain sufficient variety. Collecting such sufficiently representative and wide-ranging data may be difficult, costly, and/or time consuming. Besides, data-driven models are black-box models, meaning that their internals are unknown for users. A black-box model may provide sufficient estimation accuracy but may be limited in providing a detailed understanding of the different parameters and its behavior in terms of energy and IAQ prediction compared to a white-box physical model.

### **2.1.1.3 Hybrid models**

Hybrid methods (“grey box”), that offer a combination of physics-based and data-driven prediction models, thereby have been proposed and applied to leverage the advantages and minimize the disadvantages of both methods. In black-box data-driven models, a large amount of input

variables/features are usually required by the statistical algorithms to increase the predictive accuracy, of which some parameters and equations are physically interpretable and can be better described through physical models (Amasyali and El-Gohary 2018). It retains a part of physical meaning for the prediction. The advantages of one method remove the drawbacks of the other one. There are several strategies to apply hybrid models in energy and IAQ estimation. The hybrid methods can be used to estimate some physical parameters (based on statistical methods from available datasets), which are sometimes unavailable or less reliable in conventional physics-based models (Foucquier et al. 2013). RC network models with genetic algorithms are usually used to implement this strategy. Another common strategy is to utilize statistics to implement a learning model describing the building/occupant behavior as it is the most variable and determinant factor for predicting building energy use or IAQ (Foucquier et al. 2013). This learning model is designed from a learning basis built from a physical approach. A third strategy consists in using data-driven methods in fields where physics-based models are not effective and accurate enough (Foucquier et al. 2013).

Many studies have attempted to couple the data-driven models with the physics-based models (usually reduced-order physical models such as RC thermal network models). For example, Dong et al. (B. Dong et al. 2016) developed a hybrid model that coupled a data-driven model and a RC thermal network model for predicting the energy consumption in residential buildings and compared its prediction performance to other statistical models. Ouaret et al. (Ouaret et al. 2018) developed a hybrid model of using the historical formaldehyde concentration data as the only input of the model and integrating with the physical airflow models to forecast the indoor formaldehyde

concentrations in an office. Studies revealed that hybrid models can usually perform better than conventional physical models or data-driven models (Amasyali and El-Gohary 2018).

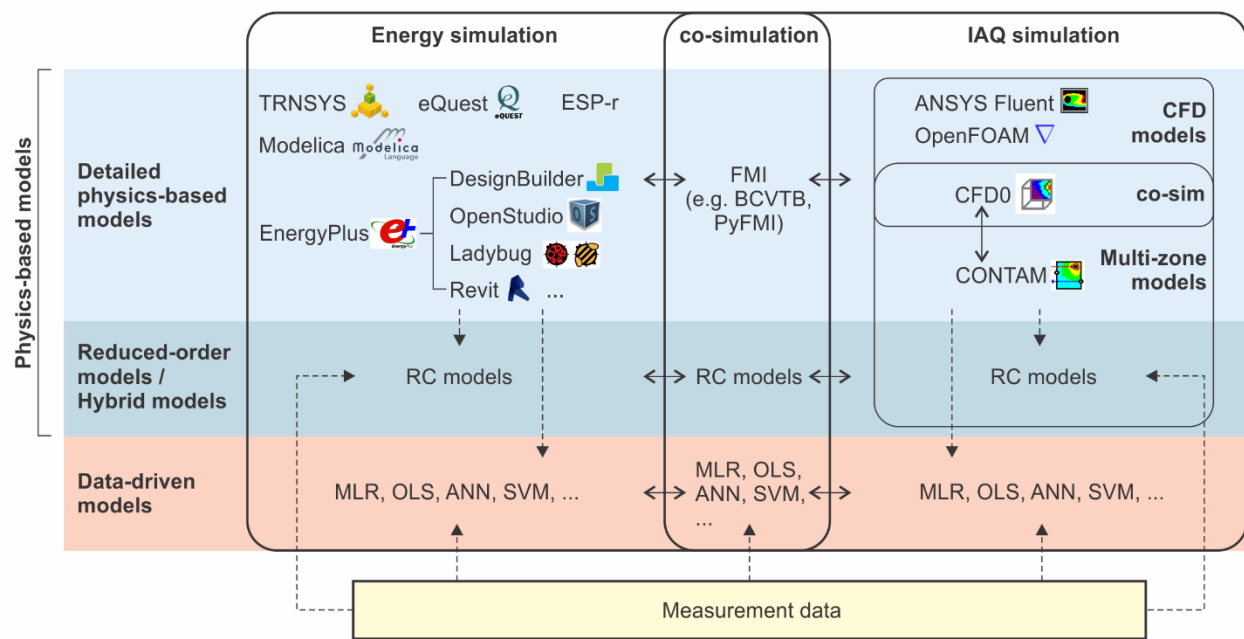


Figure 2-1. Schematic of existing BPS tools/models (S. Wang and Xu 2006).

#### 2.1.1.4 Thermal comfort assessment models

Thermal comfort and infection risk in buildings can be calculated based on the thermal and IAQ variables estimated by BPS tools/models. Some building energy simulation programs such as EnergyPlus can directly generate and export thermal comfort metrics as its outputs. Thermal comfort is usually quantified through metrics like predicted mean vote (PMV) and predicted percentage of dissatisfied (PPD), depending on the operative temperature, relative humidity, air speed, occupant's metabolic rate and clothing level. They can be calculated through methods introduced in ASHRAE Standard 55 (ASHRAE 2020a). The standard regulates the recommended

thermal comfort level in indoor spaces. To comply ASHRAE 55, the recommended thermal limit on the 7-point scale of PMV is between -0.5 and +0.5. PPD varies depending on where the occupant is in the space, and it should not exceed 20% per standard. Another standard, ISO Standard 7730 (ISO 2005), expands on this limit, giving different indoor environments ranges. It defines the hard limit as ranging between -2 and +2, for existing buildings between -0.7 and +0.7, and new buildings ranging between -0.5 and +0.5. An online thermal comfort estimation tool (CBE Thermal Comfort Tool (Tartarini et al. 2020)) has been developed by the Center for the Built Environment (CBE) of the Lawrence Berkeley National Laboratory (LBNL). It incorporates the major thermal comfort models, including the PMV, PPD, standard effective temperature (SET), adaptive models, local discomfort models, SolarCal, and dynamic predictive clothing insulation (Tartarini et al. 2020). The tool also provides dynamic and interactive visualizations of thermal comfort zones. It has several practical applications, and each year is used by more than 49,000 users worldwide, including engineers, architects, researchers, educators, facility managers and policymakers (Tartarini et al. 2020).

#### **2.1.1.5 Infection risk estimation models**

Some IAQ simulation tools such as CONTAM have the capability for simulating multizone particle transmission in indoor spaces, which has the possibility to be applied for modelling the multizone airborne transmission of viable particles. But most available IAQ performance simulation tools cannot directly perform estimations of infection risks of respiratory diseases. To address the challenges of COVID-19, many tools for estimating infection risks through airborne transmission in a single zone have been developed based on the Wells-Riley model, which is a widely used equation determining infection risks through the susceptible individual's inhalation

exposure to the suspended viable pathogen generated by the infectors (Riley et al. 1978; Stephens 2013; Sze To and Chao 2010; Wells 1955). For example, Kasibhatla et al. (Kasibhatla et al. 2020) developed a web-based tool COVID Exposure Modeler for estimating the COVID-19 infection risk from airborne transmission during classroom teaching incorporating a Monte Carlo approach. Allen et al. (Allen et al. 2020) developed a Google sheet to calculate the risk reduction by using portable air cleaners. Corsi et al. (Corsi et al. 2020) established a web-based platform named SafeAirSpaces COVID-19 Aerosol Relative Risk Estimator for evaluating the airborne infection risk and tracking the infectious particles. Dols et al. (Dols et al. 2020; William S. Dols et al. 2020) developed the web-based Fate and Transport of Indoor Microbiological Aerosols (FaTIMA) tool, which allows for single-zone analysis of airborne transmission of SARS-CoV-2 using a simplified user interface that implements CONTAM as its simulation engine.

Other tools include the Indoor Scenario Simulator developed by Riediker and Monn (Riediker and Monn 2020) for estimating virus concentration and inhaled dose in the room, the COVID-19 Ventilation Calculator created by REHVA (REHVA 2020) for estimating the effect of ventilation on COVID-19 airborne transmission, and the COVID-19 Aerosol Transmission Estimator established by Jimenez and Peng (Jimenez and Peng 2020) to evaluate infection risks in various indoor scenarios such as classrooms, supermarkets, and stadiums. Bazant et al. (Bazant 2021) developed a web-based application COVID-19 Indoor Safety Guideline that can estimate the infection risk using the Wells-Riley model and indicate the safe level of occupancy or duration time based on the input configurations. It can also assess the risk based on indoor carbon dioxide concentrations. Shen et al. (Shen, Kong, Birnkrant, et al. 2021; Shen, Kong, Dong, et al. 2021a) developed a spreadsheet-based calculator for estimating the infection risk in several space types

including ballrooms, conference rooms, bistro/cafeteria, hotel guest rooms, offices, and classrooms. The calculator can perform steady-state and stochastic simulations to evaluate the effectiveness of various mitigation strategies in reducing infection risks from baseline conditions established by ASHRAE 62.1 (ASHRAE 2019a).

However, these tools all rely on the assumption of well-mixing in a single space. Multizone airflow analysis tools, such as CONTAM, can perform viable particle transmission simulations but are unable to provide direct calculations of infection risks. As a result, currently available BPS programs cannot estimate infection risks associated with respiratory diseases in cases of multizone transmission.

#### **2.1.1.6 Research gaps**

Based on the review on existing building performance simulation tools and models, most of the reviewed tools and models are developed for engineers or researchers, which usually requires deep knowledges of building physics modelling or data science programming. These tools and models have been well applied by engineers and researchers for studying the performance of certain technologies or developing advanced strategies for energy saving and/or enhanced IAQ. However, they are inadequate, user hostile and too incomplete to be used by architects to support and inform the green building design, particularly during the early design phase (Attia et al. 2009, 2012; Riether and Butler 2008; Stine 2022). Besides, the implementation of these building performance simulation and optimization tools/models in building design tools/platforms (that are widely used by architects) is still limited. Table 2-1 shows the typical simulation and optimization tools that

were integrated in the most used architecture design software, including SketchUp, Rhino, and Revit. Most of these embedded analysis tools are established based on detailed physics-based models, e.g., EnergyPlus, Radiance, CONTAM, and OpenFOAM, while less tools rely on reduced-order physics-based models and data-driven models. Although attempts have been made to integrate building design with performance analysis, owing to the limitations and challenges of various building performance simulation tools and models mentioned earlier, barriers to integrating and practicing building performance analysis at early design phase remain high. The major research gaps of performing building performance analysis at early design phase: (1) Detailed physics-based models usually require comprehensive inputs which are likely unavailable at early design stage; (2) Performance analyses at early design phase require real-time or near-real-time responses while the simulation process of current detailed tools can be very time-consuming. (3) The performance of simulations using data-driven models relies on the selection of datasets and training variables, which may result in inaccurate estimations; (4) The co-simulation of different programs for estimating various performance metrics (e.g. energy efficiency, IAQ, thermal comfort, and infection risks) can be complicated and usually require expert background; (5) The estimation of infection risk has not been implemented in available multizone airflow simulation programs integrated with energy and IAQ assessments. Therefore, the performance analyses at early building design phase require real-time or near-real-time responses on multidisciplinary performance tasks with simplified inputs, easy-to-use user interface for architects.



Table 2-1. Common building design and performance analysis platforms and tools.

Platform	Design tools	BPS tools	BPO tools
SketchUp	SketchUp	OpenStudio ( <i>EnergyPlus, Radiance</i> )	Sefaira ( <i>parameter analysis</i> )
		Sefaira ( <i>EnergyPlus, Radiance</i> )	MOOSAS ( <i>multi-objective optimization</i> )
		designPH ( <i>simplified energy models</i> )	
Rhino	Rhino/Grasshopper	Ladybug ( <i>EnergyPlus, Radiance</i> )	Octopus ( <i>multi-objective optimization</i> )
		Honeybee ( <i>EnergyPlus, Radiance</i> )	Galapagos ( <i>multi-objective optimization</i> )
		ArchSim ( <i>EnergyPlus</i> )	
		DIVA ( <i>Daysim</i> )	
		Butterfly ( <i>OpenFOAM</i> )	
Revit	Revit	Dragonfly ( <i>UWG, CitySim</i> )	
		Energy Analysis ( <i>EnergyPlus</i> )	Insight 360 ( <i>data-driven optimization</i> )
		Insight 360 ( <i>EnergyPlus, data-driven lighting &amp; solar models</i> )	

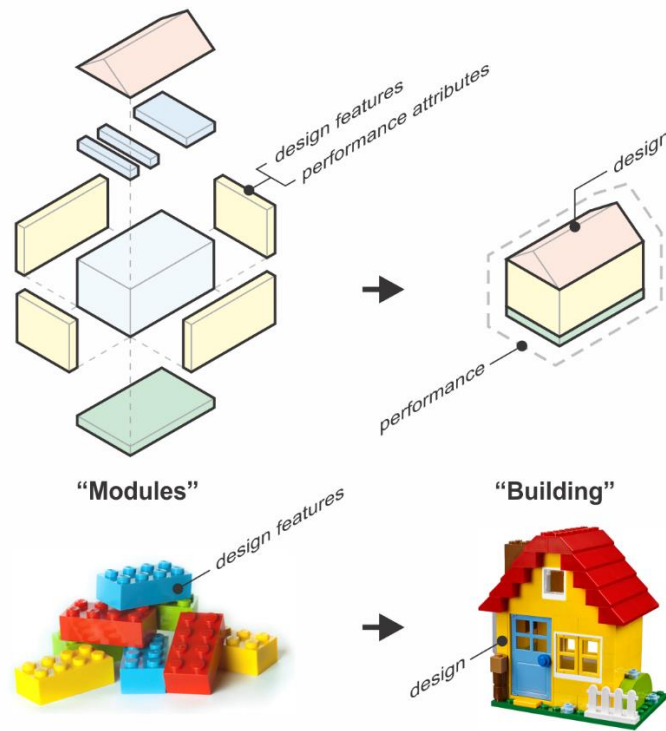
## **2.2 Methodology**

A module-based green building design and analysis platform is proposed to achieve the research goals. The platform estimates performance metrics based on the analyses on building modules with advanced models and algorithms. The platform is implemented in the Rhino Grasshopper. Some representative study cases are also introduced to be applied by the developed platform.

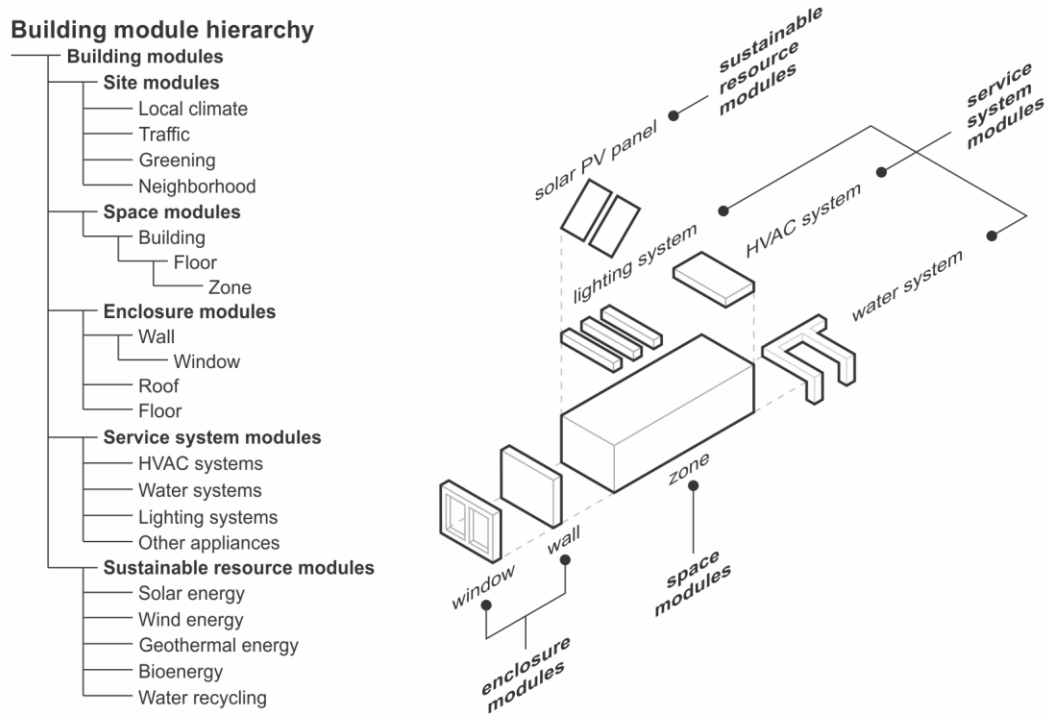
### **2.2.1 Building modules**

Every building contains a very complex set of interdependent systems comprising multiple subsystems such as the building enclosure, internal spaces/zones, and environmental service systems, which in turn consist of many components made of substantial elements. Each element has many physical parameters such as dimensions, density, specific heat, and heat conductivity for each material of a wall assembly; location, supply air temperature and flow rate of a room terminal unit of an internal space/zone; and airflow rate and pressure rise of a fan of an air handling unit of an HVAC system. A modular framework to configure these building elements can simplify and regularize various building components and systems, and provide performance metrics for data-driven modelling. Hence, building modules are defined based on building systems, subsystems, components, and elements at multiple levels, and are embedded with physical parameters and performance metrics. Building modules are like functional LEGO blocks with design features and performance attributes. These functional “LEGO blocks” can be selected, organized and integrated into a whole building of higher performance with certain algorithms. A hierarchical structure is adopted to organize the building modules that allow bottom to top aggregation across different levels (Figure 2-2). They are classified into five different types: site modules, space modules, enclosure modules, service system modules and sustainable resource modules. Site modules are

those site features related to the surrounding physical context which may affect the building performance, including local climate and weather (e.g., air temperature and solar radiance), traffic emission, surrounding vegetation (greening), and proximity to adjacent structures and building density (neighborhood). Space modules represent the building interior spaces including whole building, floors, and zones. Enclosure modules represent interior and exterior building enclosures, such as wall assembly (including window units and shading systems), ceiling and floor, which are the interfaces between zones. Service system modules include HVAC systems, lighting systems, water systems and other appliances. The service systems may consist of other specific subsystems or features. Sustainable resource modules include active renewable energy systems such as photovoltaic solar panels or active solar thermal systems, and water recycle systems.



(a) Schematic of building modules and process from modules to a building.



(b) Building module hierarchy and an example of building modules.

Figure 2-2. Building module hierarchy and schematics of building modules.

A building module is defined as an autonomous functional building block containing design and control parameters that determine its physical state and performance. It is represented as an object consisting of module parameters, state variables, interface variables, performance indicators and methods (Figure 2-3). Module parameters are further divided into design parameters and control parameters. The design parameters are those parameters determining the module's composition, geometry, and physical properties, which vary with the module types. For example, the design parameters for an enclosure wall module include the geometric dimensions, window-to-wall ratio, R-value, specific heat, and radiant absorption coefficients. Control parameters are parameters that can be adjusted during operation, such as the percent of shading for the window, and the setpoint

temperature of the HVAC system. The definition of these parameters is ready for system control and optimization. The state variables are those representing the physical state of the module, e.g., temperatures at selected locations across the wall, and the indoor concentration of a certain pollutant. The interface variables are the boundary conditions subjecting to which the module would operate. They can be either the values of the state variables at the module's boundaries or the flux related to the gradient of the state variables at the boundary, e.g., the temperature and heat flux at the boundary for building enclosure modules. The design features are usually used as the inputs for physics-based modelling.

Performance indicators are used to quantify the performance metrics of the module, such as energy saving, IAQ, thermal comfort, and risk mitigation effectiveness, in terms of its improvement potential over a reference building module per local standard or best practice. The performance metrics directly calculated through simulation, such as energy consumption, pollutant concentration and PMV/PPD, can be used to quantify the building performance. For energy analysis, the energy saving percentage relative to the reference case is used as the performance indicator. For IAQ analysis, the change of disability-adjusted life years (DALYs) to typical indoor air pollutants of a certain technology to the reference case is calculated to quantify the IAQ improvement of the studied technology. For thermal comfort analysis, the change of PMV compared to the reference case is estimated. For infection risk analysis, the infection probability changes of using a specific technology compared to the reference case is calculated as the performance indicator. The detailed definition and calculation of different performance metrics are introduced in Section 2.1.3.

The performance of a green building module or whole building may vary across regions under different climate and weather conditions. Therefore, the green performance of each building module of the target building can be evaluated by comparing its performance metrics to a local reference building. A local reference building is defined as a building, with its construction complying with the local standards/codes or adopting the local best practice. The local best practice of building design can be accessed through the local building documents, reports, cases, or available datasets, e.g., the Building Performance Database (LBNL 2011). When the local best practice is unavailable or does not meet the local building standards and codes, then the local standards and codes will be adopted to define the reference building. Many standards or codes have developed reference criteria for various types of buildings. For example, DOE (U.S. DOE 2020) has developed commercial reference buildings of 16 building types that represent approximately 70% of the commercial buildings in the U.S., according to a report published by the National Renewable Energy Laboratory (NREL) (Deru et al. 2011). The reference buildings provide complete descriptions for whole building energy analysis using EnergyPlus. The Pacific Northwest National Laboratory (PNNL) modified the DOE commercial prototype building models (U.S. DOE 2013) to accommodate the ASHRAE Standard 90.1 (ASHRAE 2019c) and International Energy Conservation Code (IECC) (ICC 2009). The PNNL commercial prototype buildings (U.S. DOE 2013) cover all DOE reference building types (with the exception of supermarkets), and also an additional prototype representing high-rise apartment buildings, resulting in 16 commercial building types in 19 climate locations. For residential buildings, PNNL uses two base prototypes to simulate single-family detached houses and multi-family low-rise apartment buildings (U.S. DOE 2013). These prototypes are then modified to accommodate four different heating system types and four foundation types typically found in new residential

construction. For single-family houses in the U.S., NREL has developed the benchmark building for each climate zone (i.e. Building America B10 Benchmark (Wilson et al. 2014)), which is consistent with the 2009 IECC (ICC 2009). Besides, ASHRAE standards (ASHRAE 2018, 2019b) defined the design criteria like envelope construction and ventilation requirement for single-family house in each climate zone, which should also be satisfied by the local reference house. The IEA Annex 68 project developed a detailed procedure along with an example for defining a local reference building for both IAQ and energy performance evaluation (Zhenlei Liu et al. 2017; Qin and Zhang 2020). Baseline settings of public buildings in terms of airborne infection risk estimation and IAQ evaluation were also presented recently (Shen, Kong, Dong, et al. 2021a). For a module's performance, the percentage of improvement over the local reference building due to the application of the green technology used in the module will be calculated. The definition of building modules, particularly the module parameters, is like the building components defined in the EnergyPlus input data files (IDFs). But the building modules defined by this study contain performance indicators and can provide interactive performance feedback, which is a big difference from IDFs. All the building systems, components or elements at any level can be modularized following this modular framework and structure. The improvement of building performance compared to the reference case will be calculated as the metrics to indicate the performance and effectiveness of certain building module.

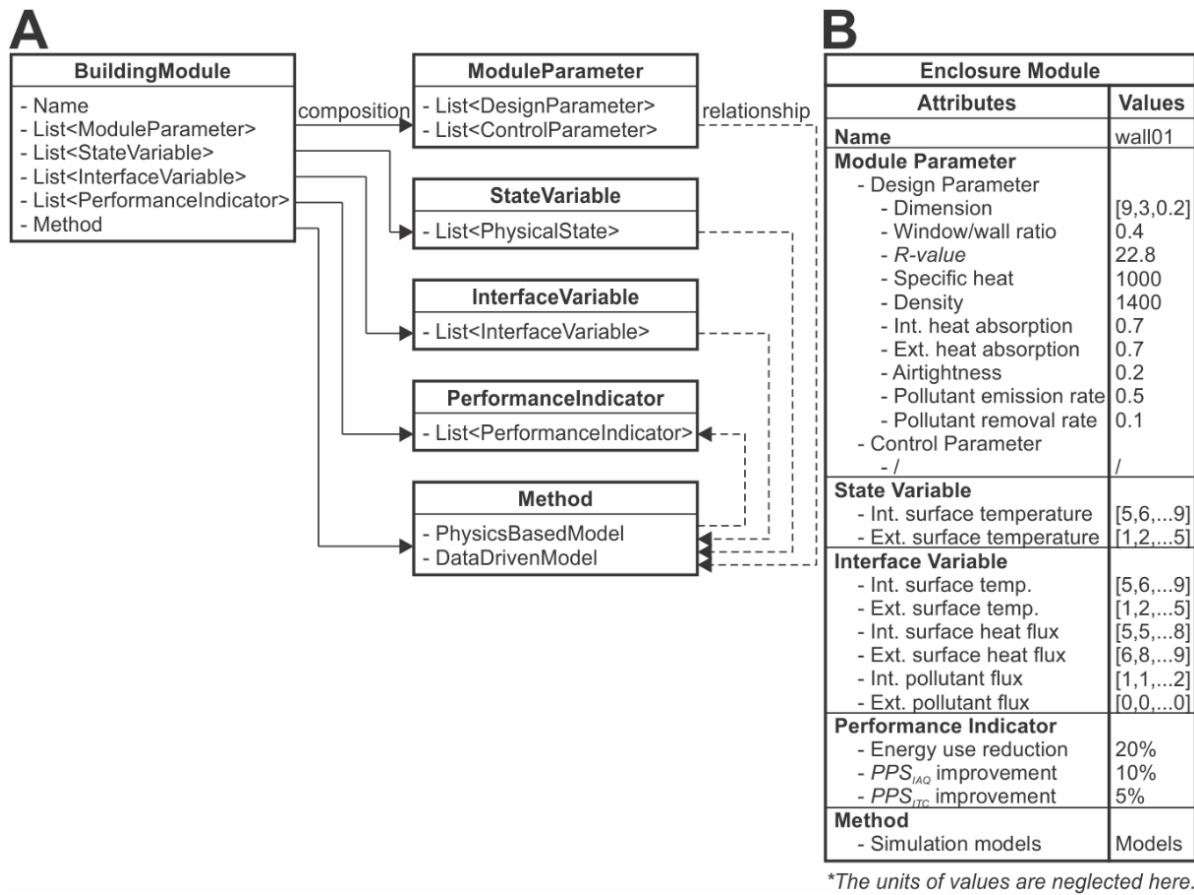


Figure 2-3. (a) Structure and composition of a building module and (b) an example of an enclosure module (wall module).

### 2.2.2 Reduced-order physics-based models

Physics-based modeling has been adequately developed over the past decades and it uses elaborated physical functions of thermodynamics, fluid dynamics and mass transfer to perform calculation precisely, and usually includes simplified modeling and detailed comprehensive modeling (ANSYS 2021; DOE 2017; NIST 2018). Due to lack of comprehensive parameters of building characteristics in early design phase and the considerable simulation process time, comprehensive/detailed physics-based modelling is usually not practical and unfavorable for



early-phase building design and performance analysis. Reduced-order physics-based models of energy and IAQ equations with certain assumptions and simplifications are more suitable for early-phase building performance analysis, which can simplify inputs through the modular approach and get (near) real-time modelling response (Rahman 2019; Xu and Wang 2007). Reduced-order resistance-capacitance (RC) thermal or air network models have been applied to simulate the thermal, air and mass transfer and balance in buildings, which are also used in this study (Braun and Chaturvedi 2002; Hao 2019; D. W. Kim et al. 2013; Z. Li et al. 2021; Mirakhorli and Dong 2016; O'Neill et al. 2010; O'Neill and Narayanan 2014; Ogunsola et al. 2014; Rahman 2019; S. Wang and Xu 2006; Xu and Wang 2007; K. Zheng et al. 2016). The RC networks, including 3R2C and 2R2C models, are used to derive a set of first order differential equations representing the building thermal, air, and mass transfer and balance. An example of RC network model for simulating thermal balance is shown in Figure 2-4. Previous studies have shown that the RC network can describe building physics in good accuracies (Hao 2019; D. W. Kim et al. 2013). Therefore, the RC network is applied to describe the thermal, air, and mass transfer and balance in this study. The used physical models in this study include thermal balance models, air balance models, and pollutant mass balance models, as well as the equations for determining the heating and cooling load, energy use, thermal comfort, occupant exposure to pollutants and health impact metrics, infection risk through airborne transmission, and cost assessment. The models are usually called nodal models or multi-zone models or lumped models, which only uses one node to represent a space or component. For example, in IAQ and pollutant transmission simulation, the indoor air is typically assumed to be well-mixed.

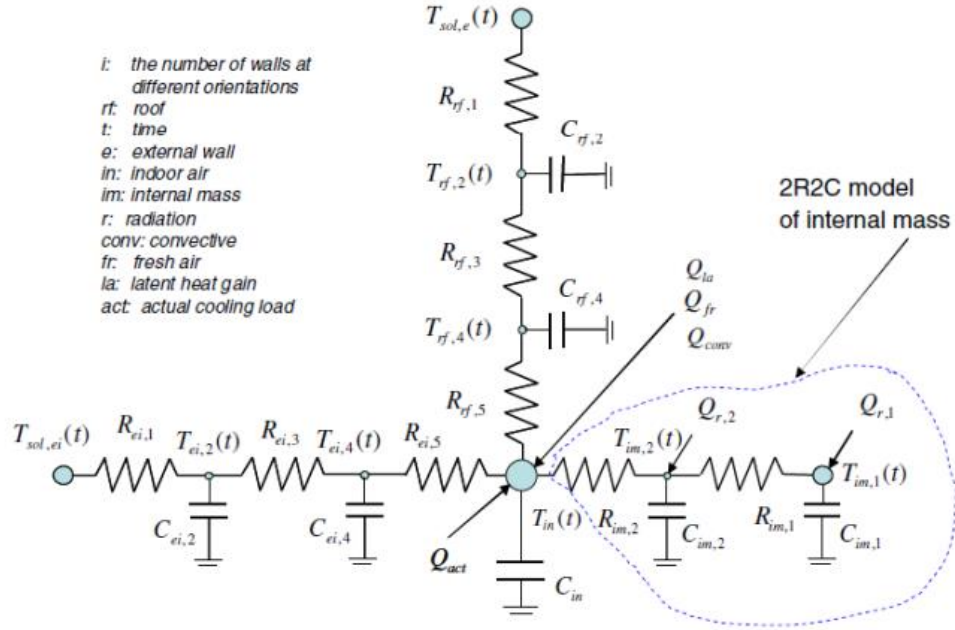


Figure 2-4. An example of RC network model for thermal modelling (S. Wang and Xu 2006).

### 2.2.2.1 Thermal models

Heat transfer in a building can be illustrated by Figure 2-5. Based on the RC thermal network, the energy balance equation for building envelopes has been applied to get the state-space model. The values of resistance and capacitance in the model represent the thermal resistance and thermal capacitance of the building, which depend on the building thermal properties. The RC network for thermal transfer and balance modelling is illustrated in Figure 2-6. In general, the resistances in the RC network cover outside convection resistances, wall conduction resistances, inside convection resistances, and window thermal resistance. The thermal mass effects are represented by the capacitances in the RC model. The network also includes the external and internal heat gains/losses that apply on external or internal thermal nodes. The  $R_s$ ,  $C_s$ , and heat gain or loss on each node ( $q_s$ ) are input parameters that are known for modelling, while air temperatures and

surface temperatures are the state variables that need to be calculated. Outdoor air temperature and ground temperature are the boundary conditions for the thermal network. In the present work, the Energyplus weather (EPW) files are adopted to use the dry air temperature and ground temperature as the boundary for modelling.

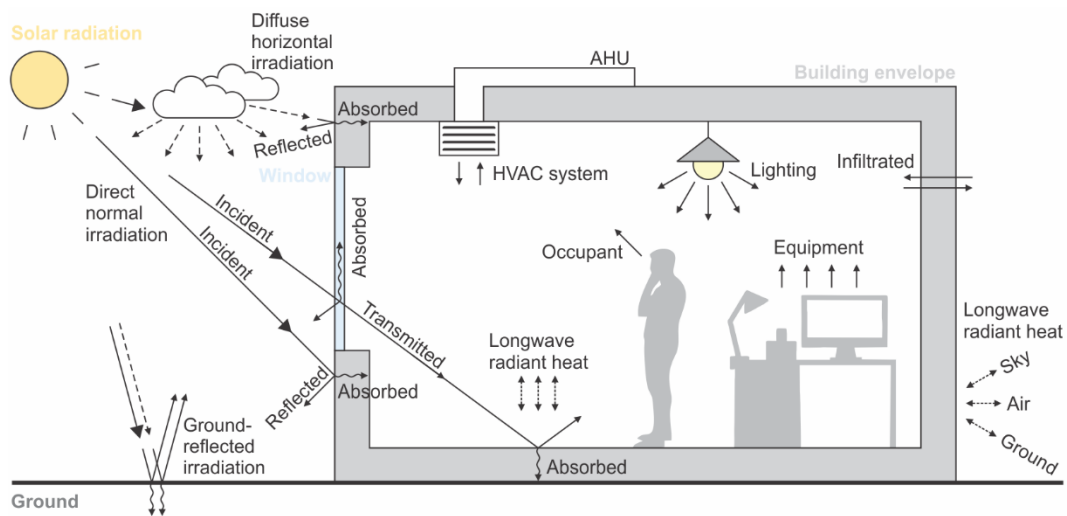


Figure 2-5. Schematic of heat transfer in a building.

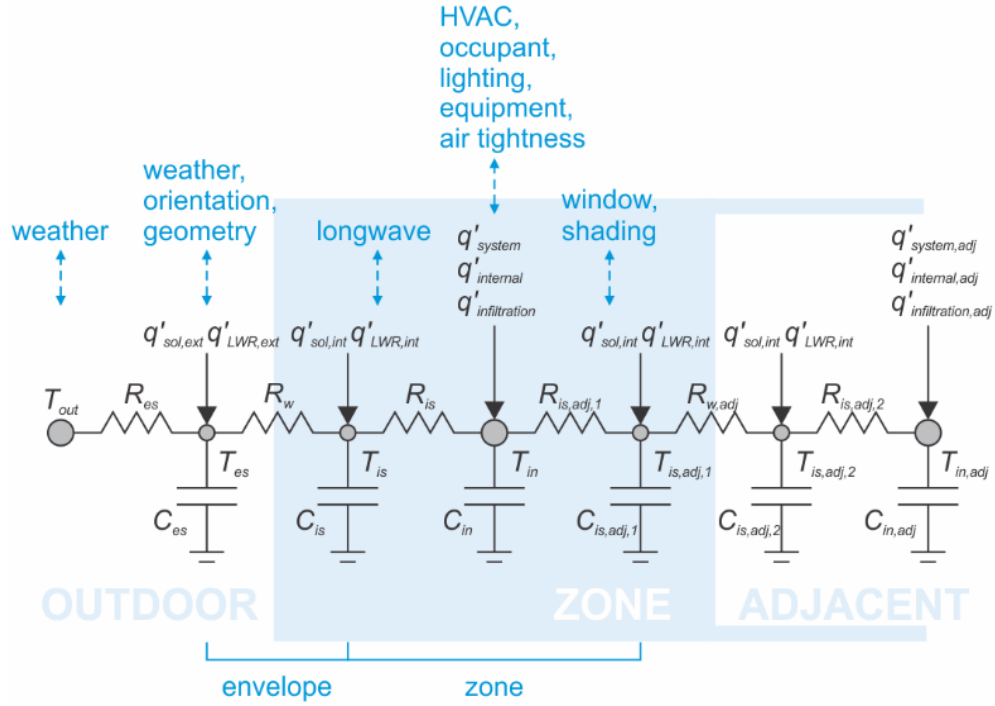


Figure 2-6. Schematic of the simplified thermal balance models in RC network.

As shown in Figure 2-7, the air temperature in each zone (room) of the network can be represented as a space node, and each envelope (external/internal wall/window) can be represented by two surface nodes (one for each side). Each envelope module can be represented by 3 Rs and 2 Cs, including the resistance of envelope ( $R_w$ ), the resistance between internal surface and interior air ( $R_{is}$ ), and resistance between external surface and exterior air ( $R_{es}$ ), and two equivalent capacitances that equally represent the half of the thermal mass of the envelope ( $C_{is}$  and  $C_{es}$ ). For each envelope, a 3R2C model is established. For an individual room, a bunch of 3R2C models are constructed and connected with each other through the space node (room air). For a building with multiple rooms, the 3R2C models of different rooms are connected by identical space nodes. Therefore, the complexity of the RC network increases with the increase of zones/rooms and

envelopes of the building. For a single room with 6 walls and no windows or other openings, the RC network typically contains 6 3R2C models connected by the space node, resulting in 1 indoor air node, 5 outdoor air nodes, 1 ground node, and 12 envelope surface nodes, which means 13 Cs (12 surface Cs and 1 indoor air C) and 18 Rs in total. Figure 2-7 shows a schematic of the RC network for a simplified two-floor building with 6 rooms (no windows). A total of 6 space nodes, 3 ground nodes, 19 outdoor air nodes, and 58 envelope surface nodes are presented. For transient thermal modelling, three types of models can be established, including the thermal models on exterior surface nodes (Eq. 2-1), the models on interior surface nodes (Eq. 2-2), and the models on space/air nodes (Eq. 2-3):

$$C_{es} \frac{dT_{es}}{dt} = \frac{T_{out}-T_{es}}{R_{es}} - \frac{T_{es}-T_{is}}{R_w} + q'_{sol,out} + q'_{rad,out} \quad (2-1)$$

$$C_{is} \frac{dT_{is}}{dt} = \frac{T_{es}-T_{is}}{R_w} - \frac{T_{is}-T_{in}}{R_{is}} + q'_{sol,in} + q'_{rad,in} \quad (2-2)$$

$$C_{in} \frac{dT_{in}}{dt} = \frac{T_{is}-T_{in}}{R_{is}} - \frac{T_{in}-T_{is,adj,1}}{R_{is,adj,1}} + q'_{system} + q'_{internal} + q'_{infiltration} \quad (2-3)$$

where  $T$  is temperature,  $C$  is thermal capacity (for thermal balance models),  $R$  is thermal resistance (for thermal balance models),  $q'$  is heat flow, and  $t$  is time. For subscripts,  $es$  is exterior surface,  $is$  is interior surface,  $w$  is wall,  $out$  is outdoor or outside,  $in$  is indoor or inside,  $adj$  is adjacent zone,  $sol$  is solar heat, and  $rad$  is longwave radiant heat.

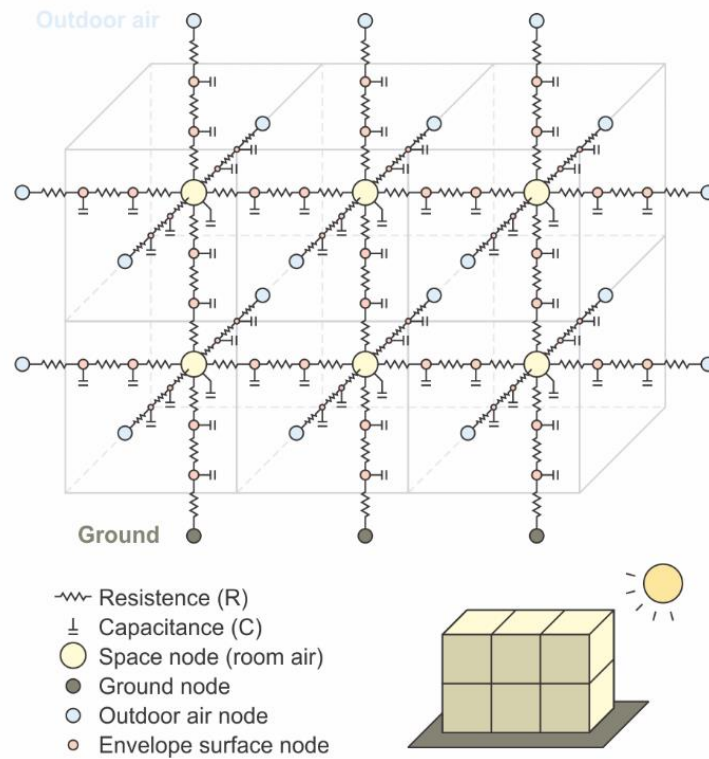


Figure 2-7. A schematic of the RC network for a two-floor building.

The thermal resistance of the building envelope module in the model is related to but not identical to the R-value of the wall assembly. It depends on the building geometry (thickness and area) and the thermal conductivity:

$$R_w = \frac{R\text{-value}}{A} = \frac{L}{k_w \cdot A} \quad (2-4)$$

where  $L$  is thickness of envelope,  $k_w$  is thermal conductivity. The thermal resistance near the envelope surfaces (interior and exterior) determines the thermal transfer between the envelope surface and the indoor or ambient air. It depends on the thermal convection near the surface as given by

$$R_{is} = h_{is}A \quad (2-5)$$

$$R_{es} = h_{es}A \quad (2-6)$$

where  $h_{is}$  is the convective heat transfer coefficient on the interior surface,  $h_{es}$  is the convective heat transfer coefficient on the exterior surface. The convective heat transfer coefficient is related to the airflow velocity near the surface. Substantial research has gone into the formulation of models for estimating the convection coefficient on building surfaces, such as the simple combined model, the TARP approach, and the MoWiTT approach (EnergyPlus 2022b). In this study, the simple combined model is applied. The simple algorithm uses surface roughness and local surface windspeed to calculate the heat transfer coefficient (EnergyPlus 2022b). The basic equation used is:

$$h = D + E \cdot v_z + F \cdot v_z^2 \quad (2-7)$$

where  $h$  is heat transfer coefficient,  $v_z$  is local wind speed calculated at the height above ground of the surface centroid,  $D$ ,  $E$ , and  $F$  are material roughness coefficients as shown in Table 2-2.

The local wind speed is dependent on the meteorological data and local environments like the surrounding terrain. It is usually calculated by Eq. (2-8). The wind speed profile coefficients  $\alpha_{wind}$ ,  $\delta$ ,  $\alpha_{met}$ , and  $\delta_{met}$ , are variables that depend on the roughness characteristics of the surrounding terrain. Typical values for  $\alpha_{wind}$  and  $\delta$  are shown in Table 2-3. The default value for  $z_{met}$  for wind speed measurement is 10 m above ground. The default values for  $\alpha_{met}$  and  $\delta_{met}$  are 0.14 and 270 m, respectively, because most meteorological stations are located in an open field.

$$v_z = v_{met} \left( \frac{\delta_{met}}{z_{met}} \right)^{\alpha_{met}} \left( \frac{z}{\delta} \right)^{\alpha_{wind}} \quad (2-8)$$

where  $z$  is altitude or height above ground,  $v_z$  is wind speed at altitude  $z$ ,  $\alpha_{wind}$  is wind speed profile exponent at the site,  $\delta$  is wind speed profile boundary layer thickness at the site,  $z_{met}$  is height above ground of the wind speed sensor at the meteorological station,  $v_{met}$  is wind speed measured at the meteorological station,  $\alpha_{met}$  is wind speed profile exponent at the meteorological station, and  $\delta_{met}$  is wind speed profile boundary layer thickness at the meteorological station.



Table 2-2. Coefficients to calculate material heat transfer properties.

<b>Roughness index</b>	<b><i>D</i></b>	<b><i>E</i></b>	<b><i>F</i></b>	<b>Example material</b>
1 (Very Rough)	11.58	5.894	0.0	Stucco
2 (Rough)	12.49	4.065	0.028	Brick
3 (Medium Rough)	10.79	4.192	0.0	Concrete
4 (Medium Smooth)	8.23	4.0	-0.057	Clear pine
5 (Smooth)	10.22	3.1	0.0	Smooth Plaster
6 (Very Smooth)	8.23	3.33	-0.036	Glass

Table 2-3. Coefficients to determine local wind speed.

<b>Terrain description</b>	<b>Exponent, <math>\alpha_{wind}</math></b>	<b>Boundary layer thickness, <math>\delta</math> [m]</b>
Flat, open country	0.14	270
Rough, wooded country	0.22	370
Towns and cities	0.33	460
Ocean	0.10	210
Urban, industrial, forest	0.22	370

The thermal capacitance is related to the specific heat of the material. For each envelope, two identical thermal capacitances are defined in this model. For indoor air, the capacitance depends on the specific heat of air and the room volume. They can be calculated as given by

$$C_{es} = C_{is} = \frac{\rho_w C_{p,w} AL}{2} \quad (2-9)$$

$$C_{in} = \rho_{air} C_{p,air} V \quad (2-10)$$

where  $C_{es}$  is the thermal capacitance on the exterior surface,  $C_{is}$  is the thermal capacitance on the interior surface,  $C_{in}$  is the thermal capacitance of indoor air,  $\rho_w$  is the density of wall material,  $C_p$  is the specific heat of the wall material,  $A$  is the wall surface area,  $L$  is the wall thickness,  $\rho_{air}$  is the air density,  $C_{p,air}$  is the specific heat of indoor air, and  $V$  is the room volume.

The external heat exchange between the exterior surfaces and the surrounding environment mainly includes solar radiation (or shortwave radiation) and longwave radiant heat exchange. Solar radiation is a key factor affecting building thermal environment. The incident solar radiation on a specific building envelope (exterior or interior) depends on the solar intensity and the geometry of the envelope. Some studies used the clear sky solar radiation model to estimate, which determines the angle of inclination on the surface based on the geographical location and the time of the day and of the year. In this study, the ambient solar intensity is obtained from the solar radiation data

in EPW files, including direct normal irradiation (DNI) and diffuse horizontal irradiation (DHI). Direct normal irradiation is the amount of solar radiation received directly from the solar disk on a surface perpendicular to the sun's rays, which means the incident DNI on building surfaces also depends on the angle of incidence  $\theta_{in}$  (between sun's rays and normal to the surface). Diffuse horizontal irradiation is the terrestrial radiation received by a horizontal surface which has been scattered or diffused by the atmosphere. The heat flux on the exterior envelope from the diffuse solar radiation is also dependent on the angle between the surface and the sky ( $\theta_{s-sky}$ ). Partial of DNI and DHI incident on the ground are reflected to the building envelope. The incident solar radiation (direct, diffuse, and ground-reflected) on the opaque building envelope is either reflected or absorbed by the exterior of the envelope, depending on the surface absorptance of the envelope. The incident solar radiation on the transparent building envelope (e.g., windows) can be reflected, absorbed, or transmitted, depending on the glazing characteristics. Eq. 2-11 determines the relationship of absorptance ( $\alpha$ ), reflectance ( $\rho$ ), and transmittance ( $\tau$ ) of the envelope. The transmitted solar radiation enters the indoor space and strikes interior surfaces (walls, floor, furniture, etc.). Therefore, the total heat flux of solar radiation on the exterior envelope is the summation of incident DNI ( $q'_{DNI,ext}$ ), DHI ( $q'_{DHI,ext}$ ), and the ground-reflected solar radiation ( $q'_{sol-reflected,ext}$ ), which can be determined by Eqs. 2-12 to 2-15 (DesignBuilder Software Ltd 2017a, 2017b; EnergyPlus 2022d). If the heat flux of DNI on the exterior surface calculated by Eq. 2-13 is negative, meaning that the direct solar irradiation does not strike on the exterior surface, the heat flux equals zero. The ground-reflected solar radiation depends on the ground reflectance ( $\rho_{gnd}$ ), the sun's zenith angle ( $\theta_z$ ), and the envelope surface angle ( $\theta_{s-sky}$ ). Ground reflectance is a factor between 0 and 1 that characterizes the fraction of solar radiation reflected by the ground. Some typical ground reflectance values are shown for various surface types in Table 2-4. The default

ground reflectance in this study is 0.2, representing typical ground types in urban or suburban areas (ASHRAE 2017a). For the ground with snow covered, a modifier is usually multiplied by the ground reflectance. The snow-covered ground usually has higher reflectance. Table 2-4 also shows the typical reflectance for snow-covered ground types. The heat flux on the exterior surface from reflected solar radiation also depends on the angle between the surface and the ground ( $\theta_{s-gnd}$ ).

$$\rho + \tau + \alpha = 1 \quad (2-11)$$

$$q'_{sol,ext} = q'_{DNI,ext} + q'_{DHI,ext} + q'_{sol-reflected,ext} \quad (2-12)$$

$$q'_{DNI,ext} = \alpha \cdot I_{DNI} \cdot \cos(\theta_{in}) \quad (2-13)$$

$$q'_{DHI,ext} = \alpha \cdot I_{DHI} \cdot \frac{1 + \cos(\theta_{s-sky})}{2} \quad (2-14)$$

$$q'_{sol-reflected,ext} = \alpha \cdot (I_{DNI} \cdot \cos(\theta_z) + I_{DHI}) \cdot \rho_{gnd} \cdot \frac{1 - \cos(\theta_{s-gnd})}{2} \quad (2-15)$$

Table 2-4. Reflectance of typical ground types (DesignBuilder Software Ltd 2017a).

<b>Surface Type</b>	<b>Surface Reflectance</b>
Water (large angle of incidences)	0.07
Coniferous forest (winter)	0.07
Bituminous and gravel roof	0.13
Dry bare ground	0.20
Weathered concrete	0.22
Green grass	0.26
Dry grassland	0.2 to 0.3
Desert sand	0.4
Light building surfaces	0.6
Snow-covered typical city center	0.2
Snow-covered typical urban site	0.4
Snow-covered typical rural site	0.5
Snow-covered isolated rural site	0.7

The radiant heat on exterior surfaces represents the longwave infrared radiation exchange between the surface, the ambient air, the ground, and the sky. The longwave heat exchange is a potential approach for building passive cooling during the nighttime (Craig et al. 2008; Hu et al. 2022; Okoronkwo et al. 2014). The radiation heat flux is calculated from the surface emittance (= absorptivity), surface temperature, sky, air and ground temperatures, and sky and ground view factors. It is assumed that each surface emits or reflects diffusely and is gray and opaque, meaning that the emittance is assumed to be equal to the absorptance. Besides, each surface is assumed to be at a uniform temperature and the heat flux is evenly distributed across the surface (EnergyPlus 2022b). The total longwave radiative heat flux ( $q'_{LWR,ext}$ ) is the sum of components due to radiation exchange with the ground ( $q'_{LWR,s-gnd}$ ), sky ( $q'_{LWR,s-sky}$ ), and air ( $q'_{LWR,s-air}$ ) as given by

$$q'_{LWR,ext} = q'_{LWR,s-gnd} + q'_{LWR,s-sky} + q'_{LWR,s-air} \quad (2-16)$$

Applying the Stefan-Boltzmann Law to each component yields, the calculation is represented as (EnergyPlus 2022b)

$$q'_{LWR,ext} = \varepsilon \sigma F_{gnd} (T_{gnd}^4 - T_{es}^4) + \varepsilon \sigma F_{sky} (T_{sky}^4 - T_{es}^4) + \varepsilon \sigma F_{air} (T_{out}^4 - T_{es}^4) \quad (2-17)$$

where  $\varepsilon$  is longwave emittance of the surface,  $\sigma$  is Stefan-Boltzmann constant,  $F_{gnd}$  is view factor of wall surface to ground surface temperature,  $F_{sky}$  is view factor of wall surface to sky temperature,  $F_{air}$  is view factor of wall surface to air temperature,  $T_{es}$  is outside surface temperature,  $T_{gnd}$  is

ground surface temperature,  $T_{sky}$  is sky temperature, and  $T_{out}$  is outdoor air temperature. Linearized radiative heat transfer coefficients are introduced to render the above equation more compatible with the heat balance formulation (EnergyPlus 2022b),

$$q'_{LWR,ext} = h_{r,gnd}(T_{gnd} - T_s) + h_{r,sky}(T_{sky} - T_s) + h_{r,air}(T_{air} - T_s) \quad (2-18)$$

where  $h_{r,gnd}$  is equivalent radiative heat transfer coefficient between envelope surface and ground,  $h_{r,sky}$  is equivalent radiative heat transfer coefficient between envelope surface and sky,  $h_{r,air}$  is equivalent radiative heat transfer coefficient between envelope surface and air. They can be determined through

$$h_{r,gnd} = \frac{\varepsilon\sigma F_{gnd}(T_{gnd}^4 - T_s^4)}{T_{gnd} - T_s} \quad (2-19)$$

$$h_{r,sky} = \frac{\varepsilon\sigma F_{sky}(T_{sky}^4 - T_s^4)}{T_{sky} - T_s} \quad (2-20)$$

$$h_{r,air} = \frac{\varepsilon\sigma F_{air}(T_{air}^4 - T_s^4)}{T_{air} - T_s} \quad (2-21)$$

The longwave view factors to ground and sky are calculated with the following expressions (Walton 1983):

$$F_{gnd} = \frac{1 - \cos\Phi}{2} \quad (2-22)$$

$$F_{sky} = \frac{1 + \cos\Phi}{2} \quad (2-23)$$

where  $\Phi$  is the tilt angle of the surface. The view factor to the sky is further split between sky and air radiation by a factor  $\beta$ :

$$\beta = \sqrt{\frac{1 + \cos\Phi}{2}} \quad (2-24)$$

The ground surface temperature is assumed to be the same as the air temperature. The final forms of the radiative heat transfer coefficients are shown here.

$$h_{r,gnd} = \frac{\varepsilon \sigma F_{gnd} (T_{air}^4 - T_s^4)}{T_{air} - T_s} \quad (2-25)$$



$$h_{r,sky} = \frac{\varepsilon \sigma F_{sky} \beta (T_{sky}^4 - T_s^4)}{T_{sky} - T_s} \quad (2-26)$$

$$h_{r,air} = \frac{\varepsilon \sigma F_{sky} (1 - \beta) (T_{air}^4 - T_s^4)}{T_{air} - T_s} \quad (2-27)$$

The air temperature can be obtained from the weather data (EPW file). The sky temperature ( $T_{sky}$ ) is calculated from the horizontal infrared radiation intensity ( $IR_H$ ), which represents the rate of infrared radiation emitted from the sky falling on a horizontal upward-facing surface, and is usually available in the weather data (EPW file) as well (EnergyPlus 2022a). It can be calculated through

$$T_{sky} = (IR_H / \sigma)^{0.25} - 273.15 \quad (2-28)$$

where  $T_{sky}$  is effective mean sky temperature, or sky radiative temperature, °C,  $IR_H$  is rate of infrared radiation emitted from the sky falling on a horizontal upward-facing surface, W/m<sup>2</sup>.

The solar irradiation on windows consists of three parts, including the reflected, the absorbed, and the transmitted parts. Solar heat gain coefficient (SHGC) is a measure of how much solar radiation is absorbed and transmitted through the glass. It is a commonly used metric for quantifying the shading effectiveness of a glazing material, depending on the glazing material and structure properties. The lower the SHGC, the less solar heat it transmits and the greater its shading ability. The absorbed solar heat by the glass will eventually be transferred to indoor air and other interior

surfaces and occupants through conduction, convection or longwave radiation. Windows will likely work with shading components such as blinds or curtains. The heat balance on shading devices can be very complicated (EnergyPlus 2022e). For example, the solar radiation that is reflected by a venetian blind depends on the slat angle and reflectance of the venetian blind, which is time-varying. In this study, the effectiveness of the window shading devices is simplified and integrated with the SHGC of the window. Therefore, the shading effectiveness will be represented by the equivalent SHGC of the window system. A window system with a more effective shading device has a lower SHGC factor than a window system without or with less effective shading units. The solar radiation transmitted through the window system (including glazing and shading) will be received by various interior surfaces like floor, walls, furniture and occupants. In this study, it is assumed that the transmitted radiation is evenly distributed on each interior surface.

Other internal heat gains include heat generated by electric lights, equipment, and occupants. The total radiant gains from lights must be divided into visible and thermal portions. In this study, the visible part is not calculated as the internal heat gain while the thermal part is added as the heat gain in the equation. For example, the total electric input to typical incandescent lights is converted to 10% visible radiation, 80% thermal radiation, and 10% convective gain. In contrast, the electric input to typical fluorescent lights is converted to 20% visible radiation, 20% thermal radiation, and 60% convective gain. In this study, it is assumed that 20% of lighting input energy is converted to visible part. The rest 80% of input energy is calculated as the internal heat gain. The lighting power in the indoor space can be estimated through ASHRAE 90.1 and 90.2 for different types of indoor spaces (ASHRAE 2018, 2019c). For example, the lighting power density is assumed to be 1 W/ft<sup>2</sup> (10.8 W/m<sup>2</sup>) for office spaces. For indoor equipment including computers, the internal heat gain

can be directly set up by the users. For portable air cleaners, it is assumed that 30% of the input energy is converted to the convective heat while 70% of the input energy is consumed to drive the fan. A survey showed that the median electric power of portable air cleaners is around 56W, which is applied in this study.

Table 2-5. The distribution of performance and price for 100 best-selling air purifiers in China (adapted from (B. Zhao et al. 2020a)).

	<b>P5</b>	<b>P25</b>	<b>Median</b>	<b>P75</b>	<b>P95</b>
<b>Clean air delivery rate (CADR) [m<sup>3</sup>/h]</b>	170	250	361	455	800
<b>Electric power [W]</b>	22	44	56	70	145
<b>Electric power per CADR [W/(m<sup>3</sup>/h)]</b>	0.10	0.12	0.15	0.19	0.28

Heat is generated in the human body by metabolic activities. The heat is dissipated from the body surface and respiratory tract by a combination of radiation, convection (sensible heat), and evaporation (latent heat) (EnergyPlus 2022f). Latent heat is not considered in this study. The heat released by the occupants depends on the metabolic rate, which equals 0.7 MET for sleeping, 1.0 MET for quietly seated, 1.2 MET for relaxed standing, 1.7-3.8 MET for walking/running at different speeds, and 4.0 MET and higher for heavy works (Tartarini et al. 2020). Therefore, metabolic rates in typical indoor scenarios are around 104-125W (1.0-1.2 MET) assuming a body

surface area of  $1.8 \text{ m}^2$  (Figure 2-8), in which the convective heat loss was approximately 40% of the total heat loss (Murakami et al. 2000; Y. Yan et al. 2016).

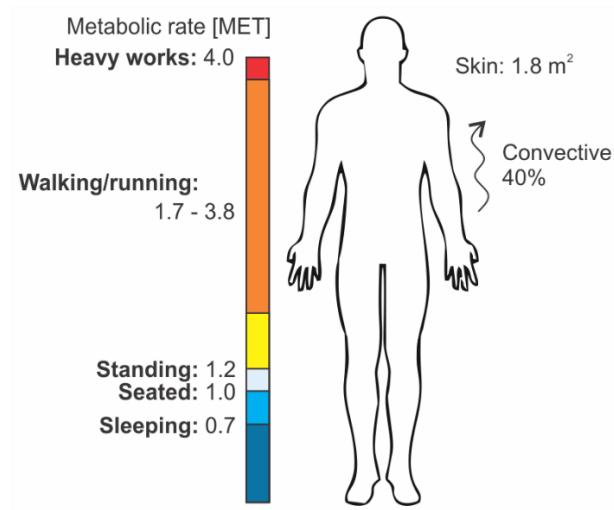


Figure 2-8. Occupant activity settings.

The actual internal heat gains from lights, equipment, and occupants also rely on their schedules and activities, which can vary greatly in different scenarios. Figure 2-9 (a) illustrates the simplified schedule of occupancy in office scenarios during weekdays and weekends. Occupant densities in different scenarios are determined by ASHRAE 62.1 (ASHRAE 2019a). The occupant density during weekends and holidays is maintained at zero. Figure 2-9 (b-c) shows the lighting and equipment schedules in office scenarios during weekdays and weekends, which are from the default settings in EnergyPlus (DOE 2017). In this study, simplified schedules are adopted to demonstrate the platform. More complicated and realistic schedules can always be input to the program to increase the representation of simulations. For example, the occupant schedules in the ASHRAE occupant behavior database (ASHRAE 2022) can be applied in the model.

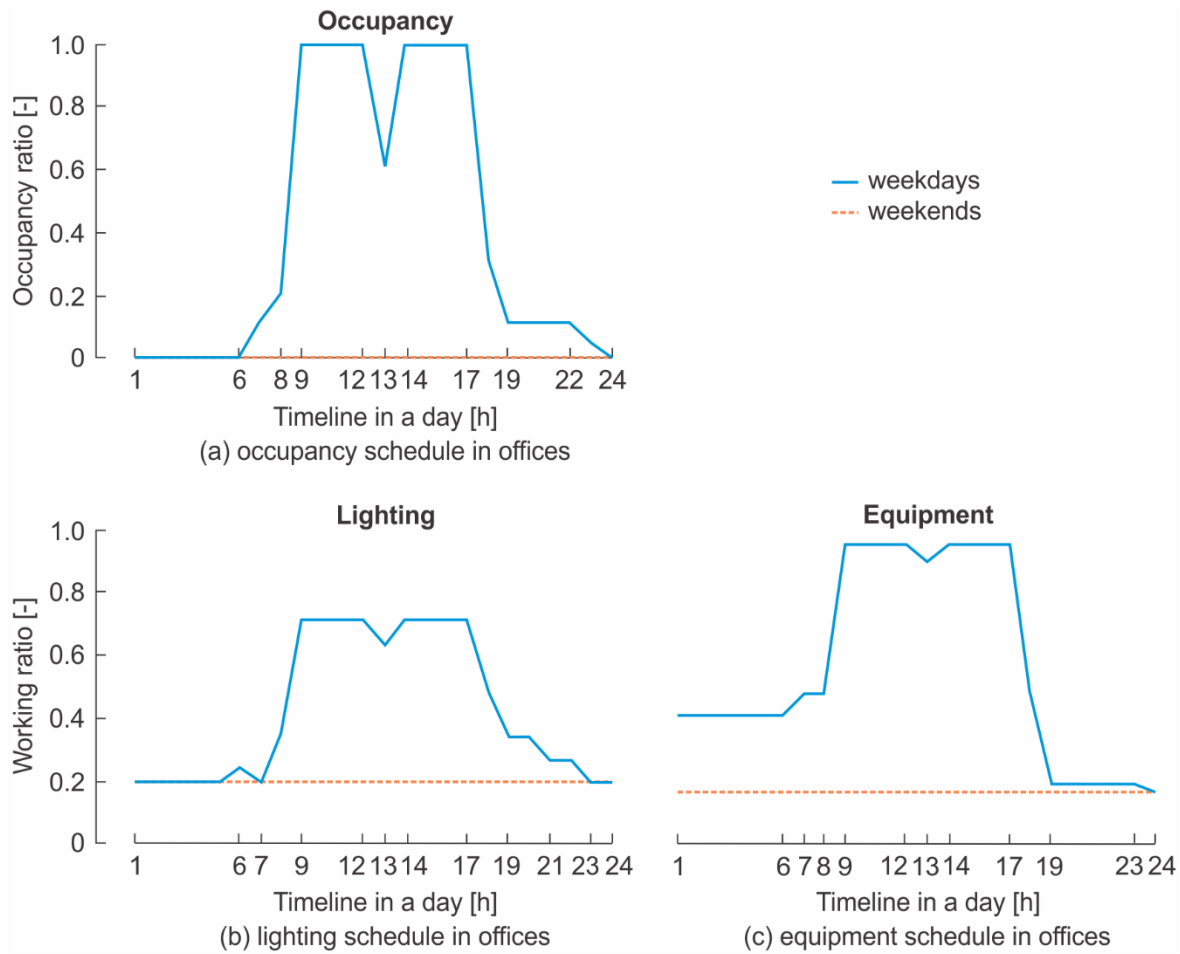


Figure 2-9. (a) Occupancy schedule, (b) lighting schedule, and (c) equipment schedule in office scenarios during weekdays and weekends.

Air infiltration through building enclosure can contribute greatly to indoor heat gain/loss as well as indoor air quality, therefore, is very significant for indoor thermal and air quality control (Crowe 2015). Eq. 2-29 represents the heat transfer through air infiltration between indoor and outdoor air. The infiltration rate can be estimated based on the pressurization tests under a specific pressure difference, usually 50 or 75 Pa, and the enclosure area. Many building standards have proposed

the suggested or mandatory baseline/reference of building enclosure air tightness (Table 2-6). ASHRAE 90.1 (ASHRAE 2019c) and IECC standards indicate that the measured air leakage through the building enclosure shall not exceed 2 L/s/m<sup>2</sup> (0.4 CFM/ft<sup>2</sup>) at 75 Pa. The 2021 International Green Construction Code (IgCC) requires the measured air leakage through enclosure shall not exceed 1.25 L/s/m<sup>2</sup> (0.25 CFM/ft<sup>2</sup>) enclosure area (at 75 Pa). The PHIUS standard requires the highest air tightness with the measured air leakage through enclosure below 0.5 L/s/m<sup>2</sup> (0.1 CFM/ft<sup>2</sup>) or 0.4 L/s/m<sup>2</sup> (0.08 CFM/ft<sup>2</sup>) at 75 Pa pressure difference. In this study, the ASHRAE baseline (2 L/s/m<sup>2</sup> or 0.4 CFM/ft<sup>2</sup>) is used as the reference air tightness level for building enclosure. The total air infiltration rate of the space equals the summation of the air infiltration rate through each exterior envelope.

$$q'_{infiltration} = C_{p,air} \cdot m'_{inf} \cdot (T_{out} - T_{in}) \quad (2-29)$$

Table 2-6. Air tightness requirements in different standards.

Standard	Building type	Air tightness target
ASHRAE 90.1 (ASHRAE 2019c)	All buildings except low-rise	2 L/s/m <sup>2</sup> (0.4 CFM/ft <sup>2</sup> )
	residential buildings	@ 75 Pa
IECC 2021 (International Code Council (ICC) 2021)	Commercial buildings	2 L/s/m <sup>2</sup> (0.4 CFM/ft <sup>2</sup> )
	Residential buildings	@ 75 Pa 3.0 ACH <sub>50</sub>

IgCC 2021 (ICC and ASHRAE 2021)	All buildings	1.25 L/s/m <sup>2</sup> (0.25 CFM/ft <sup>2</sup> ) @ 75 Pa
LEED BD+C (USGBC 2013)	Residential buildings	Varies
Net Zero Energy	All buildings	1.0 ACH <sub>50</sub>
Passive house (Passivhaus Institute 2023)	All buildings	0.6 ACH <sub>50</sub>
PHIUS+ 2015 (Salonvaara et al. 2015)	Buildings with 5+ storeys	0.5 L/s/m <sup>2</sup> (0.1 CFM/ft <sup>2</sup> ) @ 75 Pa
	All other buildings	0.4 L/s/m <sup>2</sup> (0.08 CFM/ft <sup>2</sup> ) @ 75 Pa

HVAC systems are used to condition the supply air to meet certain heating or cooling loads in the space/building. Heating and cooling loads can be estimated through the proposed models. An ideal HVAC system was set up to meet the heating/cooling loads. In this study, the energy consumption by the HVAC system mainly includes the energy use by the heating/cooling coils for conditioning the air, and the fan used to drive the air in the duct. Other energy end-use sectors also include the lighting energy use, equipment energy use, and energy uses by other facilities (e.g. portable air cleaners).

Renewable energy technologies can utilize renewable resources such as solar radiation, geothermal, and wind energy, to produce energy and reduce the net energy consumption. Typical renewable energy generation technologies are presented in later sections. Solar energy is typically the most utilized energy in buildings, through either passive (e.g., solar chimney) or active (e.g., photovoltaics panel or PV panel) utilizations. In this study, a simple PV panel example is presented. The physical model for the simple PV panel example is:

$$P_{PV} = A_{PV} \cdot f_{activ} \cdot G_T \cdot \eta_{cell} \cdot \eta_{invert} \quad (2-30)$$

where  $P_{PV}$  is the electrical power produced by photovoltaics,  $A_{PV}$  is the net area of PV surface,  $f_{activ}$  is the fraction of surface area with active solar cells,  $G_T$  is the total solar radiation incident on PV array,  $\eta_{cell}$  is the module conversion efficiency,  $\eta_{invert}$  is the DC to AC conversion efficiency. Net energy use can be estimated by considering both energy consumption and energy generation.

#### **2.2.2.2 Air and mass transfer models**

For the airflow network based on the RC approach, the room air infiltration, natural ventilation, mechanical ventilation, contaminant sources and removals are considered. Research has shown that the RC network can describe the building heat transfer and a model with higher quality can give a better estimation accuracy (Hao 2019; D. W. Kim et al. 2013). The RC thermal/airflow network models have also been applied to integrate advanced control strategies such as model predictive control (MPC) in buildings (Boodi et al. 2019; Mirakhorli and Dong 2016), or work with other statistical methods such as genetic algorithm (GA) to establish hybrid models based on



measurements or other datasets for better predicting building performance (Hao 2019; Xu and Wang 2007). In general, the reduced-order RC network models can provide fast and relatively accurate estimations on building performance with simplified inputs. Therefore, it is probably more suitable to apply reduced-order models in the early design phase as many physical information of buildings or systems required by detailed physics-based models/tools is unavailable at early design phase, which may result in poor prediction performance if failed to provide accurate inputs. Besides, the detailed physics-based models/tools usually consume more processing and simulation time, which is unfavorable for early-phase building design as there are always rapid changes to the building design at this phase, so real-time or near-real-time feedback of building performance estimation is required.

The transfer of indoor air pollutants and pathogens is illustrated in Figure 2-10. Indoor air pollutants can be originated from the outdoor air and indoor emission sources. Outdoor pollutants enter the indoor environment through infiltration or natural ventilation, as well as the air duct system. Indoor emission sources can be from interior materials, furniture, equipment, and occupant activities, depending on the nature of the pollutant species. Indoor pollutants can be removed by natural deposition on material surfaces, chemical reactions with materials and gaseous compounds, and filtered/removed by indoor air cleaning devices (e.g. portable air cleaners). Chemical reactions between compounds may also generate secondary emissions of pollutants. Pollutants can also be transmitted between different rooms or spaces. Respiratory airborne pathogens (e.g. virus-laden aerosols) are generated by respiratory activities of the infectors.

Air duct system plays an important role in pollutant and infection risk control, depending on the system design and pollutant/pathogen nature. Pollutants can be transmitted between rooms or spaces through the central HVAC system. Return air from each room gets mixed in the return air duct. A part of the return air gets recirculated and mixed with the outdoor air, filtrated or cleaned by the air cleaning equipment or filters and then supplied to the rooms/spaces again. Therefore, air pollutants may get cross-contaminated between rooms.

The mass balance equation for a specific indoor air pollutant can be presented as Eq. 2-31.

$$\begin{aligned} \frac{dC_i}{dt} = & \frac{m'_{inf,i}}{V_i} (P_i C_{out} - C_i) + \frac{m'_{hvac,i}}{V_i} C_{hvac,s} - \frac{m'_{hvac,i}}{V_i} C_i + \sum \left( \frac{m'_{adj,j}}{V_i} C_{adj,j} \right) - \frac{m'_{PAC,i}}{V_i} \eta_{PAC,i} C_i - \\ & k_{d,i} C_i - k_{a,i} C_i + k_{b,i} C_i + \frac{E_i}{V_i} - \frac{R_i}{V_i} \end{aligned} \quad (2-31)$$

where,

$$C_{hvac,r} = \frac{\sum (m'_{hvac,i} \times C_i)}{\sum m'_{hvac,i}} \quad (2-32)$$

$$C_{hvac,s} = (1 - \eta_{ACD})(1 - \eta_{filter}) [\varepsilon_{hvac,rec} C_{hvac,r} + (1 - \varepsilon_{hvac,rec}) C_{out}] \quad (2-33)$$

where,  $C_i$  is the indoor concentration of the specific pollutant in room  $i$ ,  $C_{out}$  is the outdoor concentration of the specific pollutant,  $P_i$  is the penetration factor of the pollutant through infiltration or natural ventilation,  $m'_{inf,i}$  is the air flow rate of air infiltration or natural ventilation between the outdoor air and the indoor air of room  $i$ ,  $C_{hvac,s}$  is the pollutant concentration of the supply air of the HVAC system,  $m'_{hvac,i}$  is the supply and return air flow rate of the HVAC system to room  $i$ ,  $m'_{adj,j}$  represents the air flow rate between room  $i$  and the adjacent room  $j$ ,  $C_{adj,j}$  is the pollutant concentration of the adjacent room  $j$ ,  $m'_{PAC,i}$  is the air flow rate of the portable air cleaners (PAC) in room  $i$ ,  $\eta_{PAC,i}$  is the pollutant removal efficiency of the PAC in room  $i$ ,  $k_{d,i}$  is the deposition rate of the pollutant in room  $i$ ,  $k_{a,i}$  is the first-order reaction rate of the pollutant with other gaseous compounds in room  $i$ ,  $k_{b,i}$  is the first-order generation rate of the pollutant due to reactions in room  $i$ ,  $E_i$  is the total concentration-independent emission rate of the pollutant in room  $i$ ,  $R_i$  represents the total concentration-independent removal rate of the pollutant in room  $i$ ,  $V_i$  is the volume of room  $i$ .  $C_{hvac,s}$  depends on the pollutant concentration of the system return air ( $C_{hvac,r}$ ), which is the mixture of the return air from all rooms, the recirculated air ratio of the system ( $\varepsilon_{hvac,rec}$ ), and also the outdoor pollutant concentration ( $C_{out}$ ). The pollutant in the supply air duct is also filtered and cleaned by the air cleaning equipment in the system duct, including filters and other air cleaning devices (ACD).  $C_{hvac,s}$  and  $C_{hvac,r}$  can be calculated through Eqs. 2-32 and 2-33. Although many pollutants may react with each other and produce other compounds, the coupled simulation of the chemical reactions between multiple pollutants are neglected in this study.

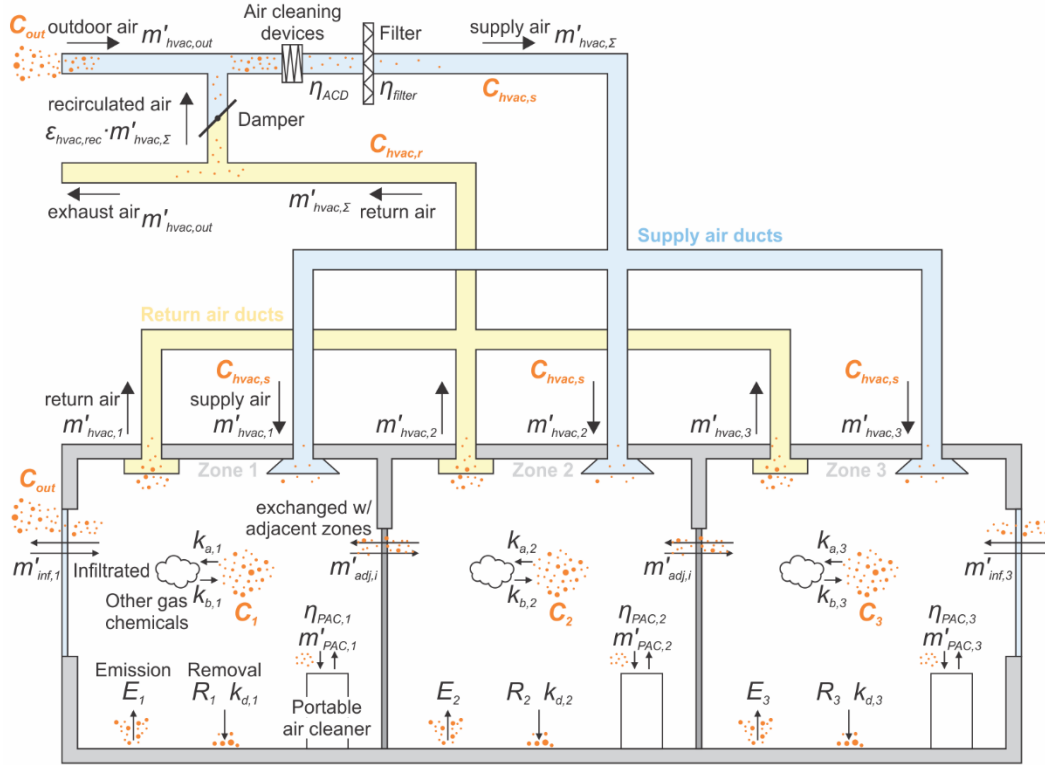


Figure 2-10. Schematic of air and mass (pollutant) flow in a building and its system.

Indoor pollutant generation can vary greatly in different scenarios. Occupant activities and indoor materials, furniture, and decorations, and specific equipment are usually the major indoor emission sources. Occupant activities can generate particles through respiratory activities and other behaviors like walking, vacuuming, cooking, smoking, and burning incense. Research (J. Qian et al. 2014) has revealed that occupant activities can contribute significantly to indoor particles. Occupant respiratory activities can also generate CO<sub>2</sub>, depending on occupant metabolic rate and age group (Eq. 2-34 and Table 2-7).

$$V'_{CO_2} = BMR \times M \times 0.000484 \quad (2-34)$$

where  $V'_{CO_2}$  is the volumetric emission rate of CO<sub>2</sub>, BMR is the basal metabolic rate (BMR), M is the metabolic rate. Ozone is usually generated by laser printer, photocopier, UV lights, ionizer, and specific air cleaners with UV or ionization units (C. Guo et al. 2019). Particle removal can be due to natural deposition or removal by air cleaners or in-duct air cleaning devices like filters. Ozone is very reactive and can be easily deposit on building materials or reacted with other chemicals (Shen and Gao 2018). Typical indoor emission sources and removals of PM<sub>2.5</sub> and ozone are shown in Table 2-8.

Table 2-7. BMR of adult age group.

<b>Age</b>	<b>BMR (MJ/day)</b>
21 to <30	8.24
30 to <40	7.83
40 to <50	8.00

Table 2-8. Typical emission sources and removal sinks and their rates for common indoor pollutants.

Pollutant	Emission		Removal	
	Source	Rate	Sink	Rate
PM2.5	Occupant activities (vacuuming) (PM0.5-20) (J. Qian et al. 2014)	300 mg/h	MERV 8 filter (Shen, Kong, Dong, et al. 2021a)	43-54% (removal efficiency)
	Occupant activities (frying) (PM0.01-1) (J. Qian et al. 2014)	540 mg/h	MERV 11 filter (Shen, Kong, Dong, et al. 2021a)	66-74% (removal efficiency)
	Occupant activities (candle burning) (PM0.01-1) (J. Qian et al. 2014)	1560 mg/h	MERV 13 filter (Shen, Kong, Dong, et al. 2021a)	82-86% (removal efficiency)
	Occupant activities (smoking) (PM0.01-1) (J. Qian et al. 2014)	600 mg/h	HEPA filter (Shen, Kong, Dong, et al. 2021a)	99.9% (removal efficiency)
	Occupant activities (incense burning) (PM0.01-1) (J. Qian et al. 2014)	240 mg/h	Natural deposition	0.25 h <sup>-1</sup> indoors (Shen, Kong, Dong, et al. 2021a)

<hr/>				
Occupant activities (walking) (PM0.8- 480 mg/h 10) (J. Qian et al. 2014)				
<hr/>				
Occupant activities (walking) (PM0.4- 54 mg/h 10) (J. Qian et al. 2014)				
<hr/>				
Occupant activities (walking) (PM2.5) 7 mg/h (J. Qian et al. 2014)				
<hr/>				
Occupant activities (walking) (PM5) (J. 24 mg/h Qian et al. 2014)				
<hr/>				
Ozone	In-duct air cleaners (e.g. ionizer, UV unit) (C. Guo et al. 2019)	62.8 mg/h	Activated carbon filter (Weschler et al. 1994)	90-95% (removal efficiency)
<hr/>				
	Ozone generator (C. Guo et al. 2019)	76.3 mg/h	Natural deposition on carpet (Shen et al. 2017; Shen and Gao 2018)	0.07 cm/s (deposition velocity)
<hr/>				

Room air cleaners (e.g. ionization-based, UV-based) (C. Guo et al. 2019)	4.6 mg/h	Natural deposition on wooden floor (Shen et al. 2017; Shen and Gao 2018)	0.02 cm/s (deposition velocity)
Photocopier (C. Guo et al. 2019)	3.3 mg/h	Natural deposition on wallpaper (Shen et al. 2017; Shen and Gao 2018)	0.03 cm/s (deposition velocity)
Laser printer (C. Guo et al. 2019)	0.8 mg/h	Natural deposition on painted wall (Shen et al. 2017; Shen and Gao 2018)	0.01-0.07 cm/s (deposition velocity)
Other small devices (C. Guo et al. 2019)	0.4 mg/h	Natural deposition on perlite ceiling tile (Shen et al. 2017; Shen and Gao 2018)	0.05 cm/s (deposition velocity)



### 2.2.3 Performance metrics

#### 2.2.3.1 Energy consumption

HVAC systems are used to condition the supply air to meet certain heating or cooling loads in the space/building. Heating and cooling loads can be estimated through the proposed models. An ideal HVAC system (variable air volume system, VAV system) was set up to meet the heating/cooling loads. In this study, the energy consumption by the HVAC system mainly includes the energy use by the heating/cooling coils for conditioning the air, and the fan used to drive the air in the duct ( $E_{fan}$ ). The energy used in the heating/cooling coil ( $E_{htg}$  and  $E_{clg}$ ) is applied to condition the intake air (the mixture of outdoor air and the recirculated air) to the temperature set up for the supply air ( $T_{hvac,s}$ ). The media fluid (refrigerant) in the refrigerator is compressed and driven by the compressor. Electric energy is used to drive the compressor and other compounds in the system. In this study, the refrigeration cycle is simplified by multiplying a conversion factor ( $\eta_{coil}$ ) to the energy use to meet the conditioning requirement by the heating/cooling coil (Eqs. 2-35 and 2-36). In this study, the conversion efficiency is assumed to be 0.9.

$$E_{htg} = C_{p,air} \cdot m'_{hvac,s} \cdot (T_{hvac,s} - T_{hvac,intake}) / \eta_{coil} \quad (2-35)$$

$$E_{clg} = C_{p,air} \cdot m'_{hvac,s} \cdot (T_{hvac,intake} - T_{hvac,s}) / \eta_{coil} \quad (2-36)$$

The fan in the air duct is used to compensate for the pressure drop along the duct and across the filter and other in-duct equipment, to provide sufficient air flow supplied to the space. The fan

power depends on the total pressure drop in the duct system, air flow rate of the duct system, and the conversion factor of the fan, which can be estimated through Eq. 2-37. The total pressure drops in the HVAC system ( $\Delta P_{hvac}$ ) include the pressure drop by the return grille ( $\Delta P_{return\ grille}$ ), return duct ( $\Delta P_{return\ duct}$ ), filter ( $\Delta P_{filter}$ ), heating/cooling coil ( $\Delta P_{coil}$ ), supply duct ( $\Delta P_{supply\ duct}$ ), and diffuser ( $\Delta P_{diffuser}$ ). The actual pressure drop along the duct is actually related to the duct length. In this study, the impact of the duct length on the pressure drop is not considered. The pressure drop of each HVAC part is presented in Table 2-9.

$$E_{fan} = m'_{hvac,s} \cdot \Delta P_{hvac} \quad (2-37)$$

$$\Delta P_{hvac} = \Delta P_{return\ grille} + \Delta P_{return\ duct} + \Delta P_{filter} + \Delta P_{coil} + \Delta P_{supply\ duct} + \Delta P_{diffuser} \quad (2-38)$$

Table 2-9. Pressure drops by different parts in the HVAC system.

Part	Pressure drops, $\Delta P$	
	in wg	Pa
Return grille	0.04	10.0
Return duct	0.08	19.9
Filter	0.12 (1 inch MERV 8 @ 1000CFM)	29.9

	0.25 (1 inch MERV 13 @ 1000CFM)	62.2
	0.50 (1 inch HEPA @ 1000CFM)	124.4
Heating/cooling coil	0.23	57.2
Supply air	0.14	34.8
Diffuser	0.03	7.5

The pressure drop across the filter relies on the filter geometry, filter level (removal efficiency), and airflow rate. It can be estimated through Eq. 2-39 (EnergyPlus 2022c), where  $K_{filter}$  is the filter pressure drop coefficient. It varies with the filter levels. For example, the  $K_{filter}$  for MERV 8, 13 and HEPA filters are  $7.18 \times 10^{-5}$ ,  $3.59 \times 10^{-5}$ , and  $1.72 \times 10^{-5}$ .

$$\Delta P_{filter} = \left[ K_{filter} + f \left( \frac{L}{D} \right) \right] \frac{(m'_{hvac,s})^2}{2 \cdot \rho_{air}} \quad (2-39)$$

Other energy end-use sectors also include the lighting energy use, equipment energy use, and energy uses by other facilities (e.g. portable air cleaners). The total energy consumption, therefore, include the HVAC use (heating, cooling and fan), lighting use ( $E_{lgt}$ ), equipment use ( $E_{eqm}$ ), and other facility use ( $E_{fac}$ ). The net energy use ( $E_{net}$ ) for building needs to consider the generation of any renewable or sustainable energy production ( $E_{pro}$ ). The net energy use then can be calculated through

$$E_{net} = (E_{htg} + E_{clg} + E_{fan}) + E_{lgt} + E_{eqm} + E_{fac} - E_{pro} \quad (2-40)$$

### 2.2.3.2 IAQ metrics

As people spend nearly 90% of their time indoors (Klepeis et al. 2001; Schweizer et al. 2007), indoor air quality (IAQ) plays a crucial role for occupant health, e.g. the indoor ozone exposure accounts for 59% (95% confidence interval: 26%-79%) of the total ozone exposure for humans (Xiang et al. 2019). IAQ is a complex problem largely because it involves many types of pollutants (Figure 2-11): (1) inorganic compounds (e.g., CO, SO<sub>2</sub>, NO<sub>x</sub>, and O<sub>3</sub>); (2) organic compounds including volatile organic compounds (VOCs, e.g. acetaldehyde, acetone, benzene, formaldehyde, naphthalene, phenol, xylenes, etc.) and semi-volatile organic compounds (SVOCs, e.g. polycyclic aromatic hydrocarbons or PAHs, di-n-butyl phthalate or DnBP, butyl benzyl phthalate or BBP, and di(2-ethylhexyl), Phthalate or DEHP); (3) radioactive gases (e.g., radon); (4) particulate matters (PM<sub>10</sub>, PM<sub>2.5</sub>, and ultrafine particles); and (5) bio-aerosols derived from virus, bacteria, fungi, protozoa, dust mites, and pollen (J. J. Zhang et al. 2022). Ozone-initiated oxidation products are also significant air contaminants including many VOCs and SVOCs, e.g. methacrolein, methyl vinyl ketone, nitrogen dioxide, acetone, 6-MHO, geranyl acetone, 4-OPA, formaldehyde, nonanal, decanal, 9-oxo-nonanoic acid, azelaic acid, and nonanoic acid.

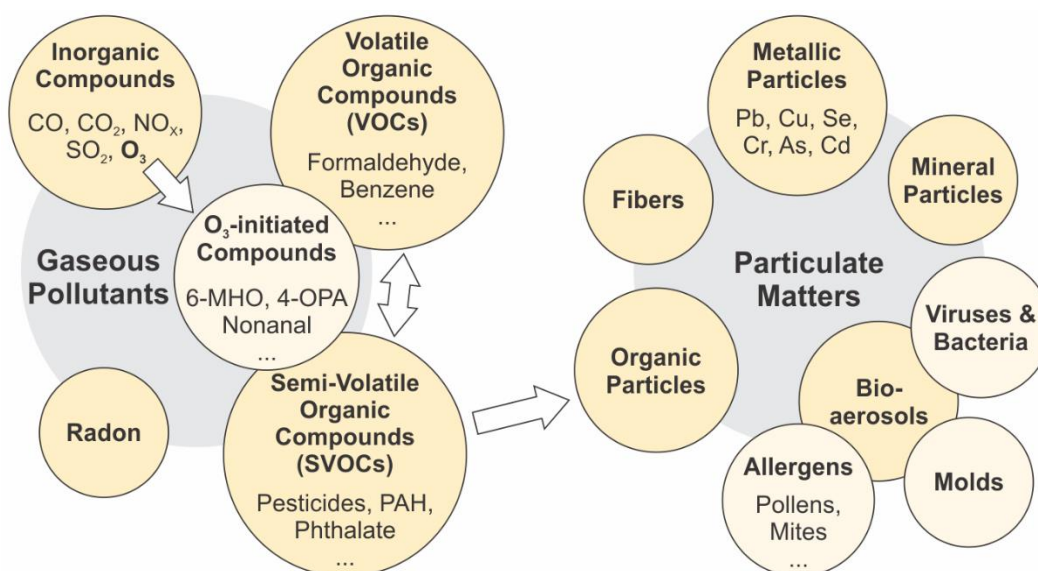


Figure 2-11. Classification of indoor pollutants (J. J. Zhang et al. 2022).

Various indoor air pollutants usually have quite different indoor sources, concentration levels, and health impacts. Inorganic gaseous compounds, such as NO<sub>x</sub>, CO and SO<sub>2</sub>, mainly originate from the ambient air that are released by traffic and industrial exhausts (U.S. EPA 2022c), and can enter buildings in the process of ventilating or by air infiltration through the building envelope (M. O. Abadie and Wargocki 2017). Ground-level ozone is a secondary pollutant and formed through chemical reactions of other compounds, specifically NO<sub>x</sub> and VOCs through photochemical reactions under certain ultraviolet (UV) light (Minnesota Pollution Control Agency 2022). Some of these compounds can also be directly emitted in indoor environments by combustion processes performed indoors, such as smoking, heaters, burning incense, cooking, etc. in the case of NO<sub>x</sub> and CO, and by printers and copiers or other ozone sources in case of ozone (M. O. Abadie and Wargocki 2017; C. Guo et al. 2019; U.S. EPA 2021a). The VOC family consists of hundreds of compounds for which emission sources are multiple and have not been completely characterized

to date. The major sources of indoor VOCs include ambient air, building materials and decoration products, occupant activities such as cleaning, cooking and combustion, and some electronic equipment like computers.

Various indoor air pollutants may have different indoor levels in different scenarios. Indoor ozone concentrations are usually below 20 ppb (Nazaroff and Weschler 2021). Measurements in approximately 2000 indoor environments show an average indoor ozone concentration of 4-6 ppb and the interquartile range of about 2-11 ppb in homes, schools, and offices (Nazaroff and Weschler 2021). In some regions with worse ambient air quality, the indoor ozone concentrations can be higher. Indoor concentrations of fine particles vary considerably, from less than  $10 \mu\text{g}/\text{m}^3$  up to thousands of  $\mu\text{g}/\text{m}^3$ , greatly depending on the indoor generation or resuspension due to occupant activities (Z. Zhou et al. 2016). The total volatile organic compounds (TVOC) levels in buildings are typically around  $300 \mu\text{g}/\text{m}^3$  (M. O. Abadie and Wargocki 2017; Logue et al. 2012). Figure 2-12 shows concentration distributions of typical air contaminants measured in the residential buildings based on a review of multiple studies in U.S., Europe, Asia, and Australia including over 20 thousand residential building cases in total (M. Abadie et al. 2019; M. O. Abadie and Wargocki 2017). Some studies revealed the most significant VOCs in offices include formaldehyde, xylene, d-limonene, hexanal, acetaldehyde, 2-ethyhexanol, and  $\alpha$ -pinene, with concentrations from as low as  $0.5 \mu\text{g}/\text{m}^3$  to over  $100 \mu\text{g}/\text{m}^3$  (M. Abadie et al. 2019; M. O. Abadie and Wargocki 2017; Logue et al. 2012).

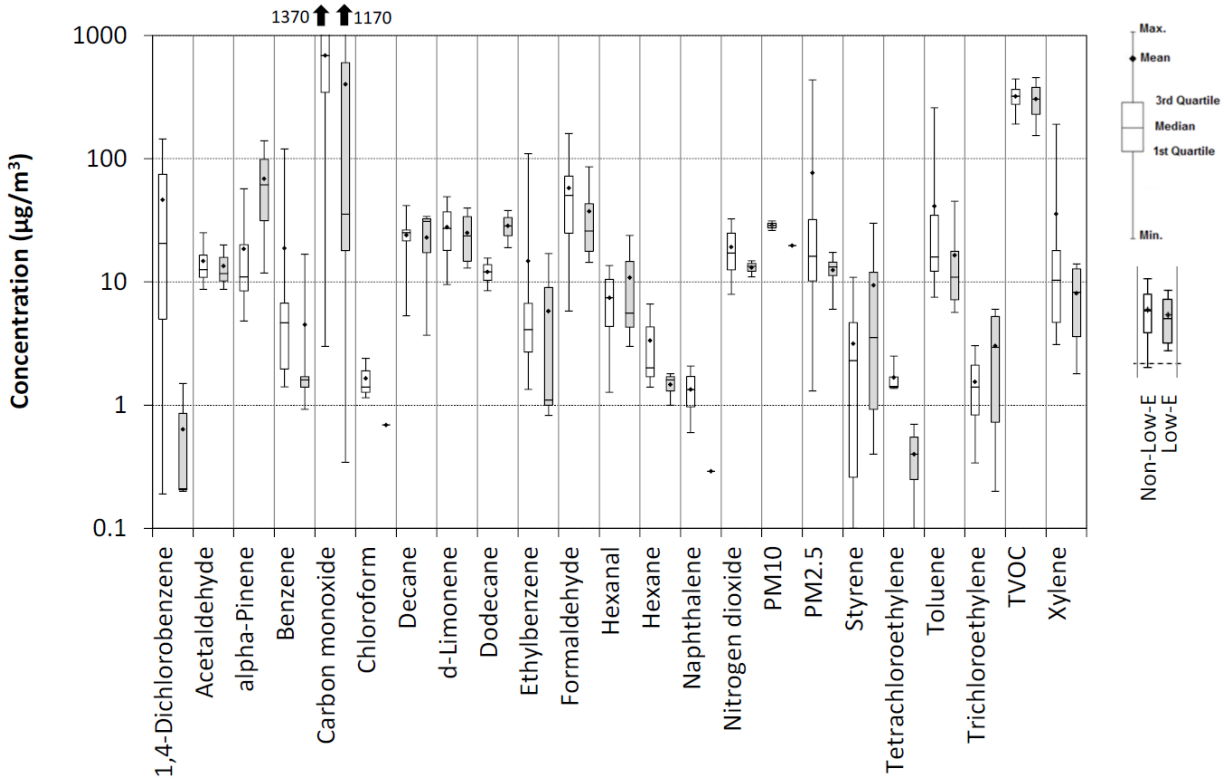


Figure 2-12. Concentration distributions of typical air contaminants measured in the residential buildings based on a review of multiple studies in U.S., Europe, Asia, and Australia (totally over 20 thousand residential buildings studied) (M. Abadie et al. 2019; M. O. Abadie and Wargocki 2017). For each pollutant, left unshaded symbols are for non-low-energy buildings and right shaded symbols correspond to low-energy residential buildings.

Most indoor air pollutants can have acute or chronic adverse health effects on people's respiratory, cardiovascular and/or reproductive systems, etc. (WHO 2022a). They can increase the morbidity and mortality associated with these diseases, reduce life quality and cause disability or even premature deaths. Some of these air pollutants are believed to be carcinogens (WHO 2022b). It has been revealed that household air pollution is responsible for an estimated 3.2 million deaths per

year in 2020 (WHO 2022a). Table 2-10 shows the health impacts of some common indoor air pollutants of concerns. Therefore, many agencies have documented standards or guidelines for indoor air pollutants, including WHO, OEHHA, and ATSDR. But the concentration limits for various pollutants are usually very different based on their typical levels and health impacts. Table 2-11 shows the concentration limits for different indoor air pollutants in standards and guidelines. Some researchers believe that there is no safe level of exposure that can be recommended for some compounds, particularly carcinogens. The indoor concentrations of them need to be maintained as low as possible. Therefore, it is not easy to quantitatively evaluate indoor air pollutants merely based on their concentrations as their indoor levels and limits may vary greatly.

Table 2-10. Health impacts of indoor air pollutants of concerns

<b>Classification</b>	<b>Compounds</b>	<b>CAS No.</b>	<b>IARC Classification <sup>a</sup> (WHO 2022b)</b>	<b>Chronic target organs <sup>b</sup> (OEHHA 2019, 2022)</b>
Inorganic gaseous compounds	CO	630-08-0		Brain (Townsend and Maynard 2002)
	SO <sub>2</sub>	9/5/7446		Respiratory system (ATSDR 2020)
	NO <sub>2</sub>	10102-44-0	/	Respiratory system (U.S. EPA 2021b)



	O3	10028-15-6		Respiratory system; Cardiovascular system (Shen and Gao 2018)
	Radon	10043-92-2	1 (Radon-222)	Respiratory system  (lung cancer) (U.S. EPA 2022b; Virginia Department of Health 2022)
VOCs	Acetaldehyde	75-07-0	2B	Respiratory system
	Acetone	67-64-1		Reproductive system; Skin; Alimentary system (liver); Kidney; Nervous system (New Jersey Department of Health 2015)
	Acrolein	107-02-8	2A	Respiratory system

Benzene	71-43-2	1	Hematologic system; Nervous system; Development
1,3-butadiene	106-99-0	1	Reproductive system
1,4-Dichlorobenzene	106-46-7	2B	Nervous system; Respiratory system; Alimentary system (liver); Kidney
Dichloromethane	75-09-2	2A	Cardiovascular system; Nervous system
Ethylene glycol	107-21-1		Respiratory system; Kidney; Development
Formaldehyde	50-00-0	1	Respiratory system
n-Hexane	110-54-3		Nervous system
Naphthalene	91-20-3	2B	Respiratory system
Phenol	108-95-2	3	Alimentary system (liver); Cardiovascular

				system; Kidney; Nervous system
	Propylene	115-07-1	3	Respiratory system
	Tetrachloroethylene	127-18-4	2A	Alimentary system (liver); Kidney
	Toluene	108-88-3	3	Acquired color vision impairment (dyschromatopsia)
	1,1,1- trichloroethane	71-55-6	2A	Nervous system
	Trichloroethylene	79-01-6	1	Nervous system; Eyes
	Xylene, total	108-83-3 (m) 95-47-6 (o) 106-42-3 (p) (mixture: 1330-20-7)	3	Nervous system; Respiratory system; Eyes
SVOCs	PAH (Benzo[a]pyrene)	50-32-8	1	Skin; Respiratory system; Alimentary

			system (liver); Kidney; Stomach; Reproductive system (New Jersey Department of Health 2017)
Particles	PM2.5	/	Respiratory system; Cardiovascular system (U.S. EPA 2022a)
	PM10	/	Respiratory system; Cardiovascular system (U.S. EPA 2022a)

<sup>a</sup> IARC: GROUP 1: the agent may be a carcinogenic mixture for humans (proven carcinogen or certainly carcinogenic); GROUP 2A: The mentioned agents are probably carcinogenic for human beings; GROUP 2B: 272 agents appear on this list of agents probably carcinogenic to humans; GROUP 3: 508 agents appear on this list and are not classifiable as to their carcinogenicity to humans; GROUP 4: to indicate agents which are probably not carcinogenic for human beings.

<sup>b</sup> Data collected from (OEHHA 2019, 2022) except the ones indicated separately.

Table 2-11. Concentration limits for different indoor air pollutants in standards and guidelines [ $\mu\text{g}/\text{m}^3$ ] (except for Radon [ $\text{Bq}/\text{m}^3$ ])

Categories	Compounds	CAS No.	OEHHA (Cal EPA) RELs (OEHHA 2019, 2022)			WHO (WHO 2005, 2010)	ANNEX 68 ELV (M. O. Abadie and Wargocki 2017)	
			Acute	8-hour	Chronic		Long-term	Short-term
Inorganic gaseous compounds	CO	630-08-0	23000					
	SO <sub>2</sub>	9/5/7446	660			20 (24-hour)		
						500 (10-min)		
	NO <sub>2</sub>	10102-44-0	470			200 (1-hour)	20	470
						40 (annual)		
	O <sub>3</sub>	10028-15-6	180			100 (8-hour)		
	Radon	10043-92-2					200	400
VOCs	Acetaldehyde	75-07-0	470	300	140		48	/
	Acetone	67-64-1						

Acrolein	107-02-8	2.5	0.7	0.35		0.35	6.9
Benzene	71-43-2	27	3	3	No SL <sup>a</sup>	0.2	/
1,3-butadiene	106-99-0	660	9	2			
1,4-Dichlorobenzene	106-46-7			800			400
Dichloromethane	75-09-2	14,000		400			
Ethylene glycol	107-21-1			400			
Formaldehyde	50-00-0	55	9	9	100 (30-min)	9	123
n-Hexane	110-54-3			7000			
Naphthalene	91-20-3			9	10 (annual)	2	/
Phenol	108-95-2	5,800		200			
Propylene	115-07-1				3000		
Tetrachloroethylene	127-18-4	20,000		35	250		

	Toluene	108-88-3	5,000	830	420	250	/
	1,1,1-trichloroethane	71-55-6	68,000		1,000		
	Trichloroethylene	79-01-6			600	2	/
	Xylene, total	108-83-3 (m)	22,000		700		
		95-47-6 (o)					
		106-42-3 (p)					
		1330-20-7 <sup>c</sup>					
	TVOC						400
SVOCs	PAH	50-32-8				No SL	
	(Benzo[a]pyrene)						
Particles	PM2.5				10 (annual)	10	25
					25 (24-hour)		
	PM10				20 (annual)	20	50

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50 (24-hour)

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(continued)

Categories	Compounds	CAS No.	BIFMA M7.1-2010 (CDGS 2008) (ANSI/BIFMA 2007)	ATSDR <sup>b</sup> (ATSDR 2022)		
				Acute	Int.	Chronic
Inorganic gaseous compounds	CO	630-08-0				
	SO <sub>2</sub>	9/5/7446				
	NO <sub>2</sub>	10102-44-0				
	O <sub>3</sub>	10028-15-6				
	Radon	10043-92-2				
VOCs	Acetaldehyde	75-07-0	9			

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Acetone	67-64-1		71,000		
Acrolein	107-02-8		6.9	0.09	
Benzene	71-43-2	30	28.8	19.2	9.6
1,3-butadiene	106-99-0				
1,4-Dichlorobenzene	106-46-7	400	12024.6	1202.5	60.1
Dichloromethane	75-09-2	200			
Ethylene glycol	107-21-1	200	5077.3		
Formaldehyde	50-00-0	16.5	49.1	36.8	9.8
n-Hexane	110-54-3	3500			2114.9
Naphthalene	91-20-3	4.5			3.7

	Phenol	108-95-2	100			
	Propylene	115-07-1				
	Tetrachloroethylene	127-18-4	17.5	40.7	40.7	40.7
	Toluene	108-88-3	150	7537.1		3768.5
	1,1,1-trichloroethane	71-55-6	500	10912.1	3819.2	
	Trichloroethylene	79-01-6	300		2.1	2.1
	Xylene, total	108-83-3 (m)	350	8683.9	2605.2	217.1
		95-47-6 (o)				
		106-42-3 (p)				
		1330-20-7 <sup>c</sup>				
	TVOC					
SVOCs	PAH	50-32-8				
	(Benzo[a]pyrene)					

Particles	PM2.5
	PM10

(continued)

Categories	Compounds	CAS No.	Health Canada (Health Canada 2018, 2022)		WELL (Air Feature 01) (IWBI 2018)
			Long-term (24-hour)	Short-term (1-hour)	
Inorganic	CO	630-08-0	11.5	28.6	11250
gaseous compounds	SO2	9/5/7446			
	NO2	10102-44-0	20	170	
	O3	10028-15-6	40 (8-hour)		100
	Radon	10043-92-2	200		148

VOCs	Acetaldehyde	75-07-0	280	1420	
	Acetone	67-64-1	70,000		
	Acrolein	107-02-8	0.44 (IARL=0.35)	38	
	Benzene	71-43-2	No SL		
	1,3-butadiene	106-99-0	1.7		
	1,4-Dichlorobenzene	106-46-7	60		
	Dichloromethane	75-09-2	600		
	Ethylene glycol	107-21-1			
	Formaldehyde	50-00-0	50	123	22
	n-Hexane	110-54-3			
	Naphthalene	91-20-3	10		
	Phenol	108-95-2			

	Propylene	115-07-1		
	Tetrachloroethylene	127-18-4	40	
	Toluene	108-88-3	2300	15000
	1,1,1-trichloroethane	71-55-6		
	Trichloroethylene	79-01-6		
	Xylene, total	108-83-3 (m)	100	
		95-47-6 (o)		
		106-42-3 (p)		
		1330-20-7 <sup>c</sup>		
	TVOC			500
SVOCs	PAH	50-32-8		
	(Benzo[a]pyrene)			
Particles	PM2.5		No SL	15

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PM10

50

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<sup>a</sup> No SL: No safe level of exposure can be recommended (pollutant concentration should be maintained as low as possible).

<sup>b</sup> For Duration, Acute = 1 to 14 days, Intermediate = 15 to 364 days, and Chronic = 1 year or longer.

<sup>c</sup> Mixture of various xylenes.

One approach to evaluate indoor air pollutants is assessing the IAQ satisfaction on health thresholds regulated in standards and guidelines, which was proposed by this study earlier (Shen et al. 2020). However, for most pollutants, any slight increase in pollutant concentration will cause certain health effects and raise the morbidity/mortality rate, even below the threshold level. For example, it is considered to be safe by the WHO standard if PM<sub>2.5</sub> levels are below 35 µg/m<sup>3</sup> within a day and the 8-hour ozone levels are below 35 ppb (WHO 2005). But every 10 µg/m<sup>3</sup> increase in daily PM<sub>2.5</sub> exposure will lead to 1% increased daily mortality rate among adults 65 years and older, and every 1ppb increase in daily ozone concentration raises the daily mortality rate by 0.5% (WHO 2005). Therefore, it is preferred to evaluate IAQ per health impacts directly. The DALY metric quantifies Disability-Adjusted Life Years (DALYs) due to exposure to a chemical substance and has been widely used to evaluate IAQ in recent years (T. Gao et al. 2014; K. K. Lee et al. 2020; Logue et al. 2012; Nurchis et al. 2022; Z Wang et al. 2009; Wysocka 2018). As defined by WHO (WHO 2020b), one DALY can be thought of as one lost year of "healthy" life. The sum of these DALYs across the population, or the burden of disease, can be thought of as a measurement of the gap between current health status and an ideal health situation where the entire population lives to an advanced age, free of disease and disability. The DALY method allows quantifying and comparing the health impact from various pollutants, including the various types of disease induced. The DALYs lost per incidence can be calculated by (Logue et al. 2012):

$$DALY_{disease} = YLL_{disease} + YLD_{disease} \quad (2-41)$$

where  $YLL_{disease}$  are years of life lost (YLL) due to premature death from the disease and  $YLD_{disease}$  are years of life disability (YLD), weighted from 0 to 1 depending on disease severity. The equation can be rewritten as:

$$DALYs = \frac{\partial DALY}{\partial \text{disease incidence}} \times \text{disease incidence} = \text{DALY factor} \times \text{disease incidence} \quad (2-42)$$

According to Logue et al. (Logue et al. 2012), there are two different ways of estimating DALYs lost for indoor air pollutants health impact. An intake-incidence-DALY (IND) approach is based on epidemiological data and epidemiology-based concentration-response (C-R) functions to quantify disease incidence. If the epidemiological data are not available, it is yet possible to use animal toxicity literature in order to calculate health impact via an intake-DALY (ID) method. Both ID and IND methods are accepted health impact models and, for both of them, only the annual average value of pollutant concentration is needed to estimate the population long-term impact. For criteria pollutants, including ozone, NO<sub>2</sub>, PM<sub>2.5</sub>, SO<sub>2</sub>, and CO, the IND method is usually applied. For other non-criteria pollutants (e.g. VOCs) the ID approach is usually used to calculate health impact associated with intake of non-criteria pollutants based on animal toxicity literature as shown in the work of Huijbregts et al. (Huijbregts et al. 2005). The IND approach is usually preferred as it provides more accurate estimations. But the IND approach can be used merely for pollutants with data on C-R functions in humans. Ozone was the only pollutant for which both the IND and ID approaches could be applied.



The IND approach calculates disease incidence using the C-R function (Eq. 2-43). The DALYs then are determined by disease incidence and DALY factor using Eq. 2-42. The exposure-related concentration is calculated by considering the inhalation of indoor polluted air of mean concentration  $C_{in}$  relative to a pollutant-free indoor air. Logue et al. (Logue et al. 2012) considered that, in the U.S., people spend 70% of their time in residential buildings so that the chronic exposure-relevant concentration contributed from indoor exposure was set to 70% of the indoor concentration (Eq. 2-44). Some other studies have suggested that the average time people spent indoors can exceed 80% (nearly 90%). Thus, people's exposure to indoor air pollutants may be higher than the estimation by Eq. 2-44. But in this study, the equation is still used for calculation.

$$\Delta incidence = population \times y_0 \times (1 - e^{-\beta_{exp} \times \Delta C_{exposure}}) \quad (2-43)$$

where  $y_0$  is the baseline prevalence of illness per year,  $\beta_{exp}$  is the coefficient of the concentration change,  $C_{exposure}$  is the exposure-related concentration, and population is the number of persons exposed (it is often set to 100,000 to obtain a DALY lost per 100,000 persons). For each pollutant and outcome,  $y_0$  and  $\beta_{exp}$  vary. Table 2-12 summarizes the health end points selected and DALY loss per incidence of disease (adapted from (Fazli and Stephens 2018; Logue et al. 2012; Lvovsky et al. 2000)). Many studies performed Monte-Carlo simulations based on the statistical distribution (95% confidence interval, CI) of  $\beta_{exp}$  and DALY factor (Logue et al. 2012). It can be observed that the coefficients for PM<sub>2.5</sub> are relatively higher than the coefficients for other pollutants, especially the DALY factors (DALY per incidence) for diseases due to PM<sub>2.5</sub> exposure. Therefore, it

suggests that the exposure to PM2.5 has stronger adverse effects on people's health compared to other compounds.

$$\Delta C_{exposure} = 0.7C_{in} \quad (2-44)$$

Table 2-12. Criteria pollutant C-R function outcomes and DALYs lost per incidence (Fazli and Stephens 2018; Logue et al. 2012; Lvovsky et al. 2000).

<b>Pollutants</b>	<b>Outcome</b>	<b><math>\beta_{exp}</math> (95% CI)</b>	<b><math>y_0</math></b>	<b>DALY factor, <math>\partial</math>DALYs/<math>\partial</math>incidence (95% CI)</b>
PM2.5	Total mortality	0.058 (0.002, 0.010)	$7.40 \times 10^{-3}$	1.4 (0.14, 14)
	Chronic bronchitis	0.091 (0.078, 0.105)	$4.00 \times 10^{-4}$	1.2 (0.12, 12)
	Nontatal stroke	0.025 (0.002, 0.048)	$2.00 \times 10^{-4}$	9.5 (9.25, 9.75), 0 complication
				11.7 (11.1, 12.4), 1 complication
				13.1 (12.2, 14.0), >1 complication
CO	Asthma	0.033 (0.016, 0.050)	$1.80 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Lung disease	0.025 (0.000, 0.057)	$2.10 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)

	Dysrhythmias	0.058 (0.012, 0.102)	$2.40 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Heart failure	0.034 (0.002, 0.066)	$3.40 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
NO2	Respiratory issues	0.004 (0.000, 0.008)	$9.50 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Congestive heart failure	0.003 (0.001, 0.004)	$3.40 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Ischemic heart disease	0.003 (0.002, 0.004)	$8.00 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Respiratory illness, indicated by symptoms	0.028 (0.002, 0.053)	N/A	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
Ozone	Mortality	0.001 (0.000, 0.002)	$7.70 \times 10^{-3}$	1 (0.1, 10)

	Asthma	0.003 (0.001, 0.004)	$1.80 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Lung disease	0.003 (0.001, 0.005)	$2.10 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Respiratory infection	0.002 (0.001, 0.003)	$5.80 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
	Dysrhythmias	0.002 (0.000, 0.004)	$2.40 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)
SO2	Hospital admissions	0.002 (0.000, 0.003)	$8.00 \times 10^{-3}$	$2.64 \times 10^{-2}$ (Fazli and Stephens 2018; Lvovsky et al. 2000)

The ID approach extrapolated directly from indoor concentrations to total DALYs lost due to intake of specific pollutants. The DALYs equation is rewritten as

$$DALYs = \frac{\partial DALY}{\partial intake} \times intake \quad (2-45)$$

where *intake* is the mass of pollutant that an individual inhales over a given time period. The DALYs per intake of the pollutant in Equation 16 consists of two categories for cancer and non-cancer effects and can be calculated by

$$\frac{\partial DALY}{\partial intake} = \left( \frac{\partial DALY}{\partial intake} \right)_{cancer} \times ADAF + \left( \frac{\partial DALY}{\partial intake} \right)_{non-cancer} \quad (2-46)$$

where  $(\partial DALY / \partial intake)_{cancer}$  and  $(\partial DALY / \partial intake)_{non-cancer}$  are DALYs lost per unit of intake from carcinogenic and non-carcinogenic causes for each pollutant (year/kg) and ADAF is the age-dependent adjustment factor. The DALYs per intake from carcinogenic and non-carcinogenic causes for various pollutants are available in Huijbregts et al. (Huijbregts et al. 2005). Table 2-13 shows the data for some selected indoor pollutants of concern adapted from (Huijbregts et al. 2005), including ozone, some VOCs and SVOCs. The full table can be found in (Huijbregts et al. 2005). The age at which carcinogens are inhaled has an appreciable effect on total toxicity, and the U.S. EPA has developed ADAFs to calculate cancer health impact as a function of exposure age (U.S. EPA 2005). To align with U.S. EPA-recommended ADAFs, Logue et al. (Logue et al. 2012)

considered three age groups: < 2, 2–16, and > 16 years of age. A population-weighted average annual air intake volume and ADAF were calculated by combining age distribution of the U.S. population, age-specific inhalation rates, and time spent at home as shown in Table 2-14. The average ADAF is 1.6 for population in the U.S. Based on the estimated ADAF, the combined DALYs per intake can be determined and is shown in Table 2-14. The mass of pollutant that an occupant inhales over a given period in an indoor space can be evaluated by:

$$intake = Q_{intake} \times C_{in} \times \Delta t \quad (2-47)$$

where  $Q_{intake}$  is the volume rate of air intake ( $m^3/day$ ),  $C_{in}$  is indoor concentration ( $\mu g/m^3$ ),  $\Delta t$  is the exposure time (day), usually taken to 365 to estimate the DALYs lost per year. The ID approach also usually works with Monte-Carlo sampling to account for confidence intervals of coefficients. But this study only performs calculations using the selected median or mean values. Earlier studies have performed both approaches with Monte-Carlo sampling method, and their results revealed that there is still a large uncertainty in the number of DALY losses estimated for each pollutant by the IND and ID methods but it is the only scientifically-based method available to evaluate and compare the health impacts of the exposure to different pollutants (M. O. Abadie and Wargocki 2017; Logue et al. 2012).

Table 2-13. DALYs per intake for selected compounds (adapted from (Huijbregts et al. 2005)).

CAS No.	Substance	Cancer $\partial\text{DALYs}/\partial\text{intake}$ [year/kg]	Non-cancer $\partial\text{DALYs}/\partial\text{intake}$ [year/kg]	Combined $\partial\text{DALYs}/\partial\text{intake}$ [year/kg]
10028-15-6	Ozone	$9.20 \times 10^{-1}$		$1.47 \times 10^0$
100-41-4	Ethylbenzene	$2.20 \times 10^{-4}$	$3.30 \times 10^{-4}$	$6.82 \times 10^{-4}$
100-42-5	Styrene	$3.30 \times 10^{-2}$	$8.30 \times 10^{-3}$	$6.11 \times 10^{-2}$
106-46-7	1,4-Dichlorobenzene	$1.20 \times 10^{-3}$	$1.90 \times 10^{-3}$	$3.82 \times 10^{-3}$
106-99-0	1,3-Butadiene	$3.00 \times 10^{-3}$	$7.10 \times 10^{-2}$	$7.58 \times 10^{-2}$
0107-02-08	Acrolein		$5.00 \times 10^1$	$5.00 \times 10^1$
108-88-3	Toluene	$2.20 \times 10^{-4}$	$4.70 \times 10^{-3}$	$5.05 \times 10^{-3}$
108-95-2	Phenol			
110-54-3	Hexane		$7.70 \times 10^{-3}$	$7.70 \times 10^{-3}$
50-00-0	Formaldehyde	$7.60 \times 10^{-1}$		$1.22 \times 10^0$
50-32-8	Benzo(a)pyrene	$1.10 \times 10^1$		$1.76 \times 10^1$
71-43-2	Benzene	$5.80 \times 10^{-3}$	$3.10 \times 10^{-3}$	$1.24 \times 10^{-2}$
75-07-0	Acetaldehyde	$6.40 \times 10^{-3}$	$3.20 \times 10^{-2}$	$4.22 \times 10^{-2}$



75-09-2	Methylene chloride	$1.10 \times 10^{-3}$		$1.76 \times 10^{-3}$
79-01-6	Trichloroethylene	$1.20 \times 10^{-3}$		$1.92 \times 10^{-3}$
91-20-3	Naphthalene	$1.10 \times 10^{-2}$	$6.10 \times 10^{-2}$	$7.86 \times 10^{-2}$

Table 2-14. Residential occupancy characteristics in the U.S. (Logue et al. 2012).

Age [years]	Percent of population [%]	Cancer ADAF	Percent of time indoors [%]	Air intake [m3/day]
<2	3	10	75	7
2-16	19	3	75	13
>16	78	1	69	15
Population average	/	1.6	70	14.4

The platform developed in this study uses both approaches for estimating DALYs for different air pollutants. An online tool for DALY estimation is also developed and can be accessed through (Shen 2022). Table 2-15 shows the typical indoor air pollutants in offices and their concentrations. DALYs for these pollutants are calculated using the present method. Figure 2-13 shows the DALYs per year per 100,000 population in office scenario. It suggests that PM<sub>2.5</sub>, acrolein, formaldehyde, ozone, and NO<sub>2</sub> are the significant pollutants that have stronger adverse health

effects on occupants in offices. Figure 2-14 illustrates the global DALYs per year per 100,000 population due to cardiovascular and respiratory diseases based on the data between 2000 and 2019 (WHO 2020a).

Table 2-15. Typical air pollutants in offices (concentrations are adapted from (Campagnolo et al. 2017)).

<b>Pollutant</b>	<b>Concentration [µg/m<sup>3</sup>]</b>	<b>DALYs per year per 100,000 population</b>
PM2.5 (Jones et al. 2021)	18	621
Ozone (Nazaroff and Weschler 2021)	12	7.06
NO2 (Salonen et al. 2019)	23	2.98
Benzene	1.8	0.0117
Toluene	4.5	0.0119
Ethylbenzene	1.2	0.00043
n-Hexane	1.5	0.00607
Limonene	12	0.0394
2-Butoxyethanol	2.5	0.0088
2-Ethylhexanol	4	0.00259

Styrene	0.8	0.0257
Formaldehyde	14	8.95
Acetaldehyde	6	0.0133
Acrolein	2.5	65.7
Benzaldehyde	1	0.00109

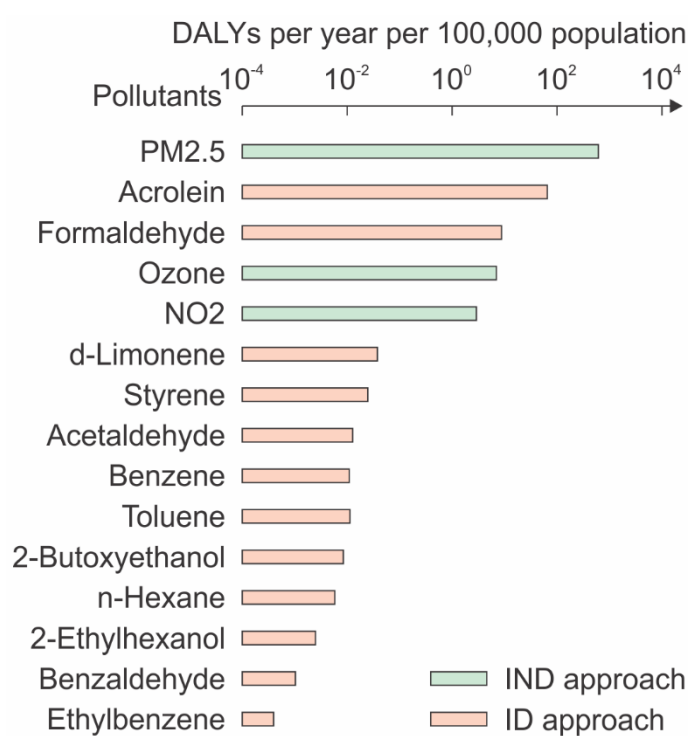


Figure 2-13. DALYs per year per 100,000 population in office scenario.

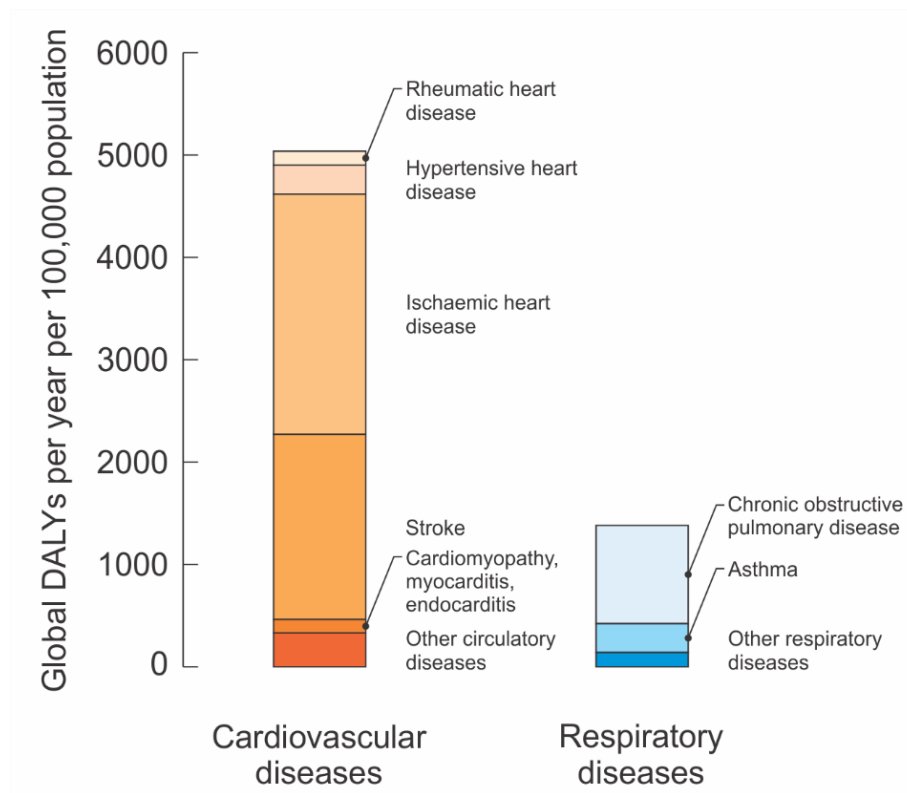


Figure 2-14. Global DALYs per year per 100,000 population due to cardiovascular and respiratory diseases (data as of 2000 to 2019 by WHO (WHO 2020a)).

### 2.2.3.3 Thermal comfort

Thermal environment is one of the main factors that influence occupants' comfort, health, work productivity and study performance (Jiang et al. 2018; Kaushik et al. 2020; Rupp et al. 2015; Tham et al. 2020). Research has revealed the strong relationship between thermal comfort and work productivity or study performance (Bueno et al. 2021; Jiang et al. 2018; Kaushik et al. 2020; Zomorodian et al. 2016). As almost a third of the workday is connected to work-related activities for adults or study-related activities for students, thermal comfort can be a key factor affecting people's work productivity and study performance (Bueno et al. 2021). Thermal environment also

influences occupant health. Exposure to both low and high indoor temperatures can increase the mortality and disorder risk relevant to respiratory and cardiovascular systems, and even morbidity under certain circumstance, particularly to the susceptible individuals like the elderly (Collins 1986; Ormandy and Ezratty 2012; Tham et al. 2020). For example, research suggests that high indoor temperatures affect aspects of human health, with the strongest evidence for respiratory health, diabetes management and core schizophrenia and dementia symptoms (Tham et al. 2020). Cold extremities and slight lowering of core temperature can induce short-term increases in blood pressure, which may be important causal factors in the increased winter morbidity and mortality due to heart attacks and strokes (Collins 1986). Symptoms of mental health disorders are also exacerbated by extreme indoor temperatures (Tham et al. 2020).

Therefore, it is essential to evaluate and control indoor thermal comfort. It is known that thermal comfort is influenced by a range of environmental and individual factors (ASHRAE 2020a; Ormandy and Ezratty 2012). The environmental factors include the air temperature, the temperature of the surrounding surfaces, the airflow velocity, the relative humidity, and the rate of air exchange (ventilation). Thermal comfort also depends on occupant activity (that varies with age, health status, and gender) and clothing. Other factors such as room crowding and body exposure to sun radiation also have influences on thermal comfort.

PMV and PPD are the most used metrics for evaluating indoor thermal comfort, which can be calculated through methods introduced in ASHRAE Standard 55 (ASHRAE 2020a). The methods in ASHRAE Standard 55 are used in this study to estimate PMV and PPD. The PMV is determined

by the heat generation and losses of the human body and the thermal sensation transfer coefficient (Eq. 2-48). Heat production on the skin surface equals metabolic heat minus the heat input from external work ( $q'_{WME}$ , which is normally around 0). Metabolic rate ( $MET$ ) is the energy produced per unit skin surface area by metabolic activities of a person, and 1 met equals  $58.15 \text{ W/m}^2$  energy produced per unit skin surface area of an average person seated at rest. Thus, the equivalent heat generation by the individual ( $q'_{MW}$ ) can be determined by Eq. 2-49. Table 2-16 shows the metabolic rates for typical indoor activities relevant to this study (selected from ASHRAE 55 (ASHRAE 2020a)). The thermal sensation transfer coefficient ( $ts$ ) is dependent on the metabolic rate (in  $\text{W/m}^2$ ) and can be calculated by Eq. 2-50.

$$PMV = ts \cdot (q'_{MW} - q'_{HL}) \quad (2-48)$$

$$q'_{MW} = q'_M - q'_W = (q'_{MET} - q'_{WME}) \cdot 58.15 \approx q'_{MET} \cdot 58.15 \quad (2-49)$$

$$ts = 0.303 \cdot e^{-0.036 \cdot q'_M} + 0.028 \quad (2-50)$$

Table 2-16. Metabolic rates for typical indoor activities relevant to this study (selected from ASHRAE 55 (ASHRAE 2020a)).

Activity	Metabolic rate	
	met	$\text{W/m}^2$

Sleeping	0.7	40
Seated (quite/reading/writing)	1.0	60
Standing, relaxed	1.2	70
Walking (0.9, 1.2, 1.8 m/s)	2.0, 2.6, 3.8	115, 150, 220
Typing	1.1	65
Lifting/packing	2.1	120
Cooking	1.6 – 2.0	95 – 115
House cleaning	2.0 – 3.4	115 – 200
Exercise	3.0 – 4.0	175 – 230

Heat losses on the skin surface include heat loss through skin ( $q'_{HL,1}$ ), heat loss by sweating ( $q'_{HL,2}$ ), latent respiratory heat loss ( $q'_{HL,3}$ ), dry respiratory heat loss ( $q'_{HL,4}$ ), heat loss by radiation ( $q'_{HL,5}$ ), and heat loss by convection ( $q'_{HL,6}$ ), which can be calculated by Eq. 2-51. Eq. 2-52 represents the heat loss through skin, depending on the body heat generation ( $q'_{MW}$ ) and water vapor pressure of indoor air ( $p_a$ ). Indoor air water vapor pressure can be determined through Eq. 2-53.

$$q'_{HL} = q'_{HL,1} + q'_{HL,2} + q'_{HL,3} + q'_{HL,4} + q'_{HL,5} + q'_{HL,6} \quad (2-51)$$

$$q'_{HL,1} = 3.05 \times 10^{-3} \cdot (5733 - 6.99 \cdot q'_{MW} - p_a) \quad (2-52)$$

$$p_a = 10 \times RH \cdot \left( 16.6536 - \frac{4030.183}{T_a + 235} \right) \quad (2-53)$$

Heat loss by sweating ( $q'_{HL,2}$ ) is determined by the sweating level of occupant under certain behaviors or activities. It is assumed that people sweat when  $> 1$  MET, which means if the metabolic rate is less than 1 met, people will not sweat and no heat will be removed by sweating ( $q'_{HL,2} = 0$ ). The heat loss by sweating when  $> 1$  MET can be calculated by:

$$q'_{HL,2} = 0.42 \cdot (q'_{MW} - 58.15) \quad (2-54)$$

Respiratory heat losses include latent heat loss ( $q'_{HL,3}$ ) and dry heat loss ( $q'_{HL,4}$ ), representing the heat removed through respiratory activities, which are affected by occupant metabolic activity and water vapor pressure (Eq. 2-55) or dry air temperature (Eq. 2-56).

$$q'_{HL,3} = 1.7 \times 10^{-5} \cdot q'_M \cdot (5867 - p_a) \quad (2-55)$$

$$q'_{HL,4} = 0.0014 \cdot q'_M \cdot (34 - T_a) \quad (2-56)$$



Heat loss by radiation ( $q'_{HL,5}$ ) and convection ( $q'_{HL,6}$ ) are through the heat transfer between outer surface temperature of the clothed body and the mean radiant temperature of surrounding surfaces or surrounding air temperature, which are calculated by Eqs. 2-57 and 2-58. The mean radiant temperature of surrounding surfaces can be determined by the surface temperature of interior surfaces and the angle factors between the individual and the surfaces. The outer surface temperature of the clothed body can be calculated as Eq. 2-59, depending on the surrounding air temperature and the clothing vapor permeation efficiency ratio ( $I_{cl}$ ).

$$q'_{HL,5} = 3.96 \times 10^{-8} \cdot f_{cl} \cdot ((T_{cl} + 273)^4 - (T_{mrt} + 273)^4) \quad (2-57)$$

$$q'_{HL,6} = f_{cl} \cdot h_c \cdot (T_{cl} - T_{in}) \quad (2-58)$$

$$T_{cl} = T_{in} + \frac{35.5 - T_{in}}{3.5 \cdot (6.45 \cdot I_{cl} + 1)} \quad (2-59)$$

$I_{cl}$  is the clothing vapor permeation efficiency ratio of actual evaporative heat flow capability through clothing to sensible heat flow capability, which is an empirical quantity and can be derived from the clothing insulation value (1 clo = 0.155 m<sup>2</sup>·°C/W). Table 2-17 shows the clothing insulation  $I_{cl}$  values for typical indoor ensembles (selected from ASHRAE 55 (ASHRAE 2020a)). However, the clothing level can be very subjective, which is difficult to determine for simulation. In this study, it is assumed that the clothing insulations in offices, schools, or other commercial/public buildings are 0.65 clo in summer and 1.00 clo in winter, and clothing

insulations in residential units are 0.55 clo in summer and 0.70 clo in winter.  $f_{cl}$  is the clothing area factor and dimensionless, depending on the clothing insulation, and can be determined by Eq. 2-60.  $h_c$  is the overall convective heat transfer coefficient between body (including clothing) and the surrounding air. The calculation of  $h_c$  is expressed as Eq. 2-61.

$$f_{cl} = \begin{cases} 1 + 1.29 \cdot I_{cl} & I_{cl} < 0.078 \\ 1.05 + 0.645 \cdot I_{cl} & I_{cl} \geq 0.078 \end{cases} \quad (2-60)$$

$$h_c = \text{Max}(2.38(T_{cl} - T_a)^{0.25}, 12.1\sqrt{v}) \quad (2-61)$$

Table 2-17. Clothing insulation  $I_{cl}$  values for typical indoor ensembles (selected from ASHRAE 55 (ASHRAE 2020a)).

<b>Clothing description</b>	<b>Garments included</b>	<b><math>I_{cl}</math> [clo]</b>
Trousers	Trousers, short-sleeve shirt	0.57
	Trousers, long-sleeve shirt	0.61
	Trousers, short-sleeve shirt, suit jacket	0.96
Skirts/dresses	Knee-length skirt, short-sleeve shirt	0.54
	Knee-length skirt, long-sleeve shirt, full slip	0.67
	Knee-length skirt, long-sleeve shirt, half slip, suit jacket	1.04

Shorts	Walking shorts, short-sleeve shirt	0.36
Sleepwear	Long-sleeve pajama tops, long pajama trousers	0.96

Some variables in the calculation of PMV are not calculated by the thermal balance models, e.g., the RH of indoor air and local airflow velocity in the indoor air. It is assumed in this study that indoor RH is maintained at 50%, which is the typical and controlled RH in indoor environments. The airflow velocity around the occupant is difficult to determine, depending on the detailed air distribution that is dependent on the ventilation unit, furniture, occupant location, and other facilities like air cleaners. The detailed air distribution and airflow velocity around the occupant usually require field measurements or CFD simulations to determine. In this study, still air is assumed in the indoor environment with the velocity of 0.1 m/s when the ventilation system is off, while the airflow velocity equals 0.5 m/s when the ventilation system is working.

After the PMV is determined, the PPD can be calculated by Eq. 2-62. The satisfied ratio is then calculated by  $1 - \text{PPD}$ . The ASHRAE 55 standard regulates the recommended thermal comfort level in indoor spaces. To comply it, the recommended thermal limit on the 7-point scale of PMV is between -0.5 and +0.5. PPD varies depending on where the occupant is in the space, and it should not exceed 20% per standard. The performance of a building module on thermal comfort satisfaction ratio is calculated compared to the reference case.

$$PPD = 100 - 95 \cdot e^{-(0.03353PMV^4 + 0.2179PMV^2)} \quad (2-62)$$

#### 2.2.3.4 Infection risk

Estimating the risk of airborne transmission in indoor environments is essential for designing control strategies to reduce infection risk and improve people's health, particularly for those diseases like COVID-19 that can cause enormous life and economic losses. The most commonly used model for estimating the airborne transmission risk in a confined space is the Wells-Riley model, which assumes the air in the space is well-mixed in steady-state (RILEY et al. 1978; Wells 1955). The model has also been extended by some researchers to include unsteady exposure (Gammaitoni and Nucci 1997) and imperfect mixing (Ko et al. 2001, 2004), and incorporate with stochastic modeling (Noakes and Sleight 2009). The infection risk determined by the Wells-Riley model depends on the susceptible individual's inhalation exposure to the viral suspended pathogen generated by the infectors (Stephens 2013; Sze To and Chao 2010).

The model has been widely adopted to estimate the airborne transmission risk of many respiratory diseases, such as influenza, tuberculosis, middle east respiratory syndrome, and measles (S. C. Chen and Liao 2008; H. Qian et al. 2009; Stephens 2013; Yates et al. 2016; Zemouri et al. 2020). Recent studies also applied the Wells-Riley model in estimating the airborne transmission risk in addressing the challenges of SARS-CoV-2 (Dai and Zhao 2020; Harrichandra et al. 2020; Pavilonis et al. 2021; Peng and Jimenez 2021; Shen, Kong, Dong, et al. 2021a; Stabile et al. 2021). The Wells-Riley model applies the C-R model for estimating the infection probability. In earlier studies, the infection possibility ( $P$ ), which can be used to estimate the new infection cases ( $N_C$ ) based on the susceptible cases ( $N_S$ ), is calculated as a function of the inhalation exposure dose

(RILEY et al. 1978), which depends on the number of infectors ( $I$ ), the emission rate of infectious quanta per infector ( $ER$ ), the equivalent fraction of infectious particle penetration through the face mask ( $\eta'_{mask}$ , including the masks by susceptible and infectious subjects), inhalation rate ( $Q_{inhale}$ ), exposure time ( $\Delta t$ ), the equivalent room air change rate ( $\Lambda$ ) and room volume ( $V$ ) as shown in Eq. 2-63. If the indoor concentration of viral particles is known, the inhalation of viral particles can be determined first based on the inhalation rate, indoor viral particle concentration, exposure time, and susceptible subjects' mask efficiency, while the equivalent indoor viral particle concentration depends on the virus emission rate and equivalent ventilation rate that is related to the ventilation effectiveness and inactivation efficiency. Then the infection probability can be determined by the C-R model. In this study, the concentration of viral particles ( $C_{vir}$ ) is first simulated using the air and mass balance models defined in above sections and the infection probability is then determined.

$$P = \frac{N_C}{N_S} = 1 - e^{-inhalation} = 1 - e^{-\eta'_{mask} \cdot Q_{inhale} \cdot C_{vir} \cdot \Delta t} = 1 - e^{-\eta'_{mask} \cdot \frac{I \cdot ER \cdot Q_{inhale} \cdot \Delta t}{V \Lambda}} \quad (2-63)$$

The equivalent air change rate ( $\Lambda$ ) represents the equivalent supply flow rate of fresh air per unit volume of the room space. It depends on the equivalent ventilation air change rate ( $\lambda_{vent}$ ), pathogen inactivation rate by ultraviolet germicidal irradiation (UVGI) system ( $k_{UV}$ ), infectious particle deposition rate ( $k_{deposition}$ ) and pathogen natural inactivation rate in the air ( $k_{inactivation}$ ):

$$\Lambda = \lambda_{vent} + f_{UV} k_{UV} + k_{deposition} + k_{inactivation} \quad (2-64)$$

The equivalent ventilation rate ( $\lambda_{vent}$ ) includes the fresh air supply rate by the HVAC system ( $\lambda_{HVAC}$ )

and standalone portable air cleaners ( $k_{AirCleaner}$ ). The original Wells-Riley model is based on the perfect-mixing assumption. However, indoor airflow patterns and mixing level are highly dependent on room configurations and air distributions. In order to evaluate the infection risk in imperfect-mixed scenarios, an additional ventilation factor ( $\epsilon_{vent}$ ) is multiplied by the ventilation rate in the model, representing the dilution efficiency in a particular location compared to the perfect mixing ventilation. It can be estimated by comparing the tracer gas concentration in the target location ( $C_i$ ) and the concentration in the exhaust air ( $C_{exhaust}$ ) through Eq. 2-65, which is similar to the zone air distribution effectiveness in ASHRAE 62.1 [22]. Some other studies performed CFD simulations to figure out the spatial concentration distribution of viruses and incorporated them with the Wells-Riley model to estimate the spatial exposure risk distribution in the space (N. P. Gao et al. 2008; H. Qian et al. 2009; Tung and Hu 2008). CFD simulations have been performed in scenarios like restaurant, classroom, grocery store, bus, car, elevator, hospital ward, and underground parking space for estimating the spatial risk distribution of SARS-CoV-2 (Dbouk and Drikakis 2021; Y. Guo et al. 2021; Y. Li et al. 2020; Han Liu et al. 2021; Mathai et al. 2021; Nazari et al. 2021; Shao et al. 2021; X. Yang et al. 2020; Z. Zhang et al. 2021). The ventilation effectiveness equals one for the perfect mixing condition. Spaces with more efficient air distribution (such as displacement ventilation) have a ventilation factor greater than one. Table 2-18 shows the ventilation effectiveness for some typical ventilation strategies (Shen, Kong, Dong, et al. 2021a; J. Zhang 2020). A similar ventilation factor was also applied incorporating with the Wells-Riley model by Sun and Zhai (Sun and Zhai 2020). The equivalent ventilation rate ( $\lambda_{vent}$ ) can be calculated by Eq. 2-66.

$$\epsilon_{vent} = \frac{C_{exhaust}}{C_i} \quad (2-65)$$

$$\lambda_{vent} = f_{HVAC}\lambda_{HVAC}\varepsilon_{vent} + f_{AirCleaner}k_{AirCleaner} \quad (2-66)$$

Table 2-18. Ventilation effectiveness of some ventilation strategies (Shen, Kong, Dong, et al. 2021a; J. Zhang 2020).

Ventilation strategy	Ventilation effectiveness
Mixing ventilation	1
Displacement ventilation	1.2 – 2
Semi-open space with partitions installed	2 – 3
Displacement ventilation with partitions installed	14 – 100
Personal ventilation	1.4 – 10

The fresh air supplied by the HVAC system ( $\lambda_{HVAC}$ ) includes the outdoor part and the recirculated part. The recirculated fresh air supply rate (Eq. 2-67) depends on the recirculated airflow rate ( $\lambda_{recirculated}$ ) and the infectious particles filtration efficiency by the filters ( $\eta_{filter}$ ).

$$\lambda_{HVAC} = \lambda_{outdoor} + \lambda_{recirculated}\eta_{filter} \quad (2-67)$$

The total supply airflow rate of the ventilation system equals to the summary of outdoor airflow rate and the recirculated airflow rate, which can be calculated by

$$\lambda_{supply} = \lambda_{outdoor} + \lambda_{recirculated} = \frac{\lambda_{outdoor}}{1-f_{recirculated}} \quad (2-68)$$

A portable air cleaner can supply additional fresh air. The infectious particle removal rate by air cleaners ( $k_{AirCleaner}$ ) can be estimated by its airflow rate ( $\lambda_{AirCleaner}$ ) and filter efficiency ( $\eta_{AirCleaner}$ ), or based on its clean air delivery rate (CADR) and room volume ( $V$ ):

$$k_{AirCleaner} = \lambda_{AirCleaner}\eta_{AirCleaner} = \frac{CADR}{V} \quad (2-69)$$

The pathogen removal rate by the UVGI system depends on the fraction of UVGI operation time ( $f_{UV}$ ) and the pathogen inactivation rate due to ultrafine (UV) irradiation ( $k_{UV}$ ). The infectious particle deposition rate ( $k_{deposition}$ ) relies on an approximate estimate of gravitational settling (Eq. 2-70) (Nicas et al. 2005), which depends on the particle diameter ( $d_p$ ) and room height ( $H$ ). The possible impacts of environmental conditions on particle deposition (L. Zhao et al. 2020) are not considered in this study.

$$k_{deposition} = \frac{0.108d_p^2(1+\frac{0.166}{d_p})}{H} \quad (2-70)$$

Virus-carrying aerosols expelled by the COVID-19 patients have been detected to be infectious in some hospital settings (Lednický et al. 2020; Santarpia et al. 2020). However, the field data are not designed to determine how long the virus can survive. Some experimental studies have also been conducted in laboratory environments to study the viability of virus-laden aerosols in the air



under different environmental conditions. The infectivity of viral aerosols can typically remain for up to hours in the air under typical indoor environment conditions. But a rapid decay was observed when the aerosols were exposed to sunlight (Dabisch et al. 2020; Schuit et al. 2020). Temperature and humidity may also influence the persistence of viable SARS-CoV-2 in aerosols. van Doremalen et al. (van Doremalen et al. 2020) observed an inactivation rate of  $0.63\text{h}^{-1}$  for SARS-CoV-2. Fears et al. (Fears et al. 2020) measured a nearly zero decay rate. Schuit et al. (Schuit et al. 2020) revealed a mean decay rate of  $0.48\text{h}^{-1}$  without sunlight. Smither et al. (Smither et al. 2020) suggested a decay rate of  $0.95\text{h}^{-1}$  in aerosols at medium humidity condition and  $0.24\text{h}^{-1}$  at high humidity condition. Dabisch et al. (Dabisch et al. 2020) observed decay rates of  $0.36\text{h}^{-1}$  and  $1.02\text{h}^{-1}$  in the environment with room temperature and no sunlight. Therefore, the typical inactivation rate of SARS-CoV-2 aerosols at typical indoor temperature and humidity is generally between 0 and  $1\text{h}^{-1}$ . Sunlight can possibly contribute greatly to the inactivation of SARS-CoV-2 (Dabisch et al. 2020; Schuit et al. 2020), but is not considered in this study. In this study, the inactivation rate of  $0.63\text{h}^{-1}$  is adopted as the natural inactivation rate for estimation.

Another significant factor that affects infection risk is the emission rate of viral particles. Quantum generation rate per infector ( $ER$ ) is a critical parameter in the Wells-Riley model. The magnitude of  $ER$  depends on disease species, infector activities (e.g. breathing, coughing) and interventions (e.g. wearing masks), and may vary significantly case by case (Buonanno, Stabile, et al. 2020; Stephens 2012). The value of  $ER$  of a COVID-19 infector is currently not well established. It is believed to be close to the  $ER$  of influenza and SARS-CoV-1 because their basic reproduction numbers ( $R_0$ ) are close (Dai and Zhao 2020; Imai et al. 2020; Khalili et al. 2020; Q. Li et al. 2020; Majumder and Mandl 2020; Read et al. 2020; S. Zhao et al. 2020). One quantum in the model

represents an infectious dose that would infect 63% of the population with the exposure per the Wells-Riley model (Riley et al. 1978). It describes the number of infectious particles in a way that implicitly includes both the number and the infectivity of virus particles (which also inherently captures particle size effects and probability of deposition in appropriate regions of the respiratory system) (Stephens 2012). The quanta emission rate has a unit of quanta per hour ( $\text{h}^{-1}$ ), and the magnitude relies on disease species, infector activities (e.g. breathing, coughing) and interventions, and may vary significantly case by case (Buonanno, Stabile, et al. 2020; Stephens 2012). The quantum generation rate can be estimated from retrospective analysis on real outbreak events if sufficient epidemiological data are available for the disease. However, the epidemiological data for emerging respiratory diseases like COVID-19 are likely insufficient especially at the early stage of outbreaks.

The viral load of the particles exhaled by the infector is closely associated with the infection risk of airborne transmission. However, it is still difficult to directly determine the viral load of the exhaled particles. Considering that exhaled particles are formed in the respiratory tract and released through respiratory activities, it is possible to derive the exhaled viral load based on the viral load in the respiratory tract, which can be detected more easily (Buonanno, Stabile, et al. 2020). A viral load model has been proposed (Eq. 2-71) to estimate the viral load emitted by a contagious subject based on the viral load in the mouth (or sputum), the type of respiratory activity (e.g., breathing, speaking, or coughing) and activity level (e.g., resting, standing, light exercise) (Buonanno, Stabile, et al. 2020). The emission rate of virus quanta ( $ER$ ) can be determined by the viral load in the sputum ( $c_v$ ), the conversion factor ( $c_i$ ), inhalation rate ( $IR$ ), and total particle volume (relies on particle number  $N_D$  and volume  $V_D$  of each size in diameter  $D$ ).

$$ER = c_v \cdot c_i \cdot IR \cdot \int (N_D \cdot dV_D) \quad (2-71)$$

The viral load in sputum has been measured in some studies. Fajnzylber et al. (2020) measured 1.8-9.0 log<sub>10</sub> RNA copies/mL in sputum with a median level of 4.4 log<sub>10</sub> RNA copies/mL. Wölfel et al. (2020) measured the average viral load in sputum of 6.85 log<sub>10</sub> RNA copies/mL, with a maximum of 9.37 log<sub>10</sub> RNA copies/mL. Yoon et al. (2020) reported the viral load in the sputum of two patients between 4.86 and 8.22 log<sub>10</sub> RNA copies/mL. Kim et al. (2020) detected the viral load in the sputum of the first two patients in South Korea roughly between 5 and 9 log<sub>10</sub> RNA copies/mL. Pan et al. (2020) collected sputum samples from 80 patients and observed a median viral load of 5.88 log<sub>10</sub> RNA copies/mL with a maximum of 11.13 log<sub>10</sub> RNA copies/mL. Zheng et al. (2020) collected 1846 respiratory samples (including sputum and saliva samples) and detected the viral load approximately between 3 and 8 log<sub>10</sub> RNA copies/mL. To et al. (2020) measured the peak viral load in the respiratory tract between 3.91 and 7.56 log<sub>10</sub> RNA copies/mL with a median of 5.29 log<sub>10</sub> RNA copies/mL for patients with mild disease, and the peak viral load between 4.27 and 7.40 log<sub>10</sub> RNA copies/mL with a median of 6.91 log<sub>10</sub> RNA copies/mL for patients with severe disease. Han et al. (2020) detected the viral load in the sputum of a patient roughly between 4 and 6 log<sub>10</sub> RNA copies/mL. It can be observed that the viral load in sputum can reach a level as high as 11 log<sub>10</sub> RNA copies/mL, while most of the existing data is approximately below 9 log<sub>10</sub> RNA copies/mL. A recent study (J. Li et al. 2021) reviewed and analyzed the existing data of viral load in sputum in literature (397 samples) and observed a normal distribution of viral load with a mean value of 4.87 log<sub>10</sub> RNA copies/mL and a standard deviation of 1.90, indicating 99% of the viral load in sputum is below 9.30 log<sub>10</sub> RNA copies/mL. Some

variants may have higher viral loads than the original strain. A study reported over 1000 times greater viral loads in oropharyngeal swabs for the Delta variant than the original viral lineage (B. Li et al. 2021).

Some studies have conducted measurements of virus distribution in indoor environments rather than directly on exhaled droplets. Stephens (2013) reviewed such studies for influenza and estimated the size-resolved distribution of pathogen using the data in the literature (W G Lindsley et al. 2010), i.e. 15% of pathogens in the 0.3-1 $\mu$ m size range, 25% in the 1-3 $\mu$ m size range, and 60% in the 3-10 $\mu$ m size range. A report from CDPH (2020) suggested the distribution of 20%, 30%, and 50% for infectious particles of SARS-CoV-2 in 0.3-1 $\mu$ m, 1-3 $\mu$ m and 3-10 $\mu$ m, respectively. It was assumed that those indoor measurements account for a combination of all human respiratory activities that occur indoors (Stephens 2013). In this study, it is assumed that 15% of virus-laden particles are in the 0.3-1 $\mu$ m size range, 25% in the 1-3 $\mu$ m size range, and 60% in the 3-10 $\mu$ m size range.

Table 2-19 shows the quantum generation rate of SARS-CoV-2 from retrospective analysis and through the viral load model in literature. In this study, the infectious quantum generation rate is estimated based on the viral load model (Eq. 2-71). Previous studies measured the viral load of COVID-19 patients and suggested that the viral load can typically reach  $10^9$  RNA copies/mL (Buonanno, Stabile, et al. 2020; Dubert et al. 2020; Han et al. 2020; J. Y. Kim et al. 2020; Pan et al. 2020; Rothe et al. 2020; Wölfel et al. 2020), which is used in this study. A reported average value of 0.02 is applied as the  $c_i$  in the model (Buonanno, Stabile, et al. 2020). The highest droplet number concentration in (Buonanno, Stabile, et al. 2020) is adopted. The droplet volume

calculation uses the geometric mean diameters for each particle size bin (i.e.,  $0.55\mu\text{m}$ ,  $1.7\mu\text{m}$ , and  $5.5\mu\text{m}$  for particles of  $0.3\text{-}1\mu\text{m}$ ,  $1\text{-}3\mu\text{m}$ , and  $3\text{-}10\mu\text{m}$ , respectively [98]). Three different activity levels are considered: sedentary and light-intensity (breathing or whispering while seated or standing), moderate-intensity (speaking while seated or standing) and high-intensity (breathing or speaking while running or doing exercises).

Table 2-19. Quanta emission rates ( $ER$ ) of SARS-CoV-2 particles in different studies.

Activity	$ER$ [ $h^{-1}$ ]	Introduction	Reference
<b>Estimated from retrospective analysis on real outbreak events</b>			
Standing + singing	970 $\pm$ 390	Skagit Valley Chorale superspreading event	Miller et al. (2020)
Standing + singing	341	Skagit Valley Chorale superspreading event	Buonanno et al. (2020)
Standing + singing	870	Skagit Valley Chorale superspreading event	Bazant et al. (2021)
Seated + vocalization	61	Guangzhou restaurant outbreak event	Buonanno et al. (2020)
Resting + breathing	45	Zhejiang tour coach outbreak event	Bazant et al. (2021)
Resting + breathing	30	Diamond Princess cruise ship outbreak event	Bazant et al. (2021)
Resting + breathing	29	Wuhan city outbreaks	Bazant et al. (2021)
Resting + breathing	185.63	Diamond Princess cruise ship outbreak event	Chen et al. (2021)
<b>Estimated using the viral load model</b>			
Resting	<1		Buonanno et al. (2020)
Intermediate	$\leq 100$	Estimated based on the viral load in the sputum	
Light activity + vocalization	>100		
Resting + breathing/whispering <sup>a</sup>	3	Estimated based on $c_v = 10^9$ RNA copies/mL and	Buonanno et al. (2020)

Standing + breathing/whispering	3	$c_i = 0.02$
Light activity + breathing/whispering	9	
Resting + speaking <sup>b</sup>	50	
Standing + speaking	56	
Light activity + speaking	142 <sup>c</sup>	

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**Estimated using statistical methods**

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Sedentary state	14-48	Estimated based on the fitting curve between $ER$ Dai and Zhao (2020b) and $R_0$ from the data of other respiratory diseases (e.g., influenza and SARS-CoV-1)
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<sup>a</sup> Breathing/whispering represents the mean value between whispering and breathing (Buonanno, Stabile, et al. 2020).

<sup>b</sup> Speaking is considered as the mean value between unmodulated vocalization and voiced counting (Buonanno, Stabile, et al. 2020).

<sup>c</sup> This value was adopted by many studies (Harrichandra et al. 2020; S. Zhang and Lin 2020).

Genetic variants of SARS-CoV-2 have been emerging and circulating around the world throughout the COVID-19 pandemic (U.S. CDC 2021d). Evidence has shown that some variants are associated with an increase in transmissibility and more severe disease, e.g. increased hospitalizations or deaths, and may also reduce the effectiveness of previous treatment measurements (U.S. CDC 2021d). At the current stage, the main SARS-CoV-2 variants of concern (VOCs) include the Alpha variant (B.1.1.7, first detected in the U.K.), the Beta variant (B.1.351, first detected in South Africa), the Gamma variant (P.1, first detected in Brazil), the Epsilon variant (B.1.427 and B.1.429, first detected in California) and the Delta variant (B.1.617.2, first detected in India). Studies have revealed that these variants generally have higher transmissibility than the original lineage, therefore, cause even more concerns about their spreading. The Alpha variant was observed 43 to 90% more transmissible than the predecessor lineage (N. Davies et al. 2021). The Beta variant was estimated to be 1.56 (95% CI 1.50-1.74) times transmissible as previously circulating lineages (Pearson et al. 2020). The Epsilon variant was detected and estimated to have an 18.6-24% increase in transmissibility (Deng et al. 2021). The latest and prevailing variant so far, the Delta variant, is responsible for over 83% of new infections in the U.S. as of July 2021 (Fry and Rapp 2021). It is highly contagious, nearly twice as transmissible as the original strain or previous variants (with an odds ratio of 1.88 (95% CI 0.95-3.76) compared with the original type (Ong et al. 2021) or an odds ratio of 1.64 (95% CI 1.26-2.13) compared to the Alpha variant (Hester Allen et al. 2021)). A recent report indicated that the viral loads of Delta infections were on average around 1000 times greater compared to the original strain of COVID-19 (B. Li et al. 2021). In many countries, the Omicron variant has become the major challenge of COVID-19. Thus, the transmission of the airborne route for the Omicron variant requires certain attention and studies. In this study, the virus generation rate is assumed to be 200 quanta/h.



Face masks provide air filtration at a personal level for wearers, which is a critical strategy for mitigating infection risk. Face masks can reduce the average emission rate by approximately 30%, 50% and 95% with cloth, surgical and N95 masks, respectively (Mueller et al. 2020). Konda et al. (Konda et al. 2020) measured the mask filtration efficiency for particles in different diameters (Table 2-20). The particle-size weighted removal efficiencies of different masks can be estimated based on the assumed infectious particle size distribution. The particle-size-weighted efficiency is around 32%, 44% and 95% for cloth, surgical and N95 masks, respectively. This study uses the present Wells-Riley model for estimating the infection risk in indoor spaces. An online tool is also developed based on the proposed model and can be accessed through: (Shen, Kong, Birnkrant, et al. 2021).

Table 2-20. Mask filtration efficiency for particles with different sizes.

Mask	Particle removal efficiency $\eta_{filter}$ [%]			Particle-size-weighted <sup>c</sup>
	0.3-1 $\mu$ m	1-3 $\mu$ m	3-10 $\mu$ m	
Cloth (cotton/silk, with gap) <sup>a</sup>	27	33	34	32-33
Surgical (with gap) <sup>a</sup>	41	44	45	44
N95 <sup>b</sup>	95	95	95	95

<sup>a</sup> Average value of the data measured in (Konda et al. 2020).

<sup>b</sup> Assuming 95% for all size ranges.

<sup>c</sup> Monte Carlo approach is implemented that adopts uniform probability distribution of particle sizes, i.e. 10-20% in 0.3-1 $\mu$ m, 20-30% in 1-3 $\mu$ m, and remaining 50-70% in 3-10 $\mu$ m.

## 2.2.4 Simulation methods and implementation

### 2.2.4.1 Model linearization

State-space matrices (SSM) are created to solve the RC network models of thermal and mass balance equations, respectively. The SSM representation is a mathematical model of a physical system as a set of input, output and state variables related by first-order linear differential equations, which is widely used for solving control problems (X. Chen et al. 2015; Fontenot et al. 2021; Tang and Wang 2019). Therefore, the SSM approach is suitable for solving the problems presented in this study. The SSM for thermal and mass balance models with  $p$  inputs ( $p_1$ ,  $p_2$ , and  $p_3$ ),  $q$  outputs and  $n$  state variables can be represented as Eq. 2-72.  $x$  is the state vector representing the state variables in this study such as air and surface temperatures and pollutant concentrations,  $x \in R^n$ .  $u$ ,  $v$ , and  $w$  are the input (or control) vectors and are usually represented as a single vector as  $u$  in other studies. In this study, to better describe and distinguish different parameters, the input vector is divided into three vectors,  $u \in R^{p_1}$ ,  $v \in R^{p_2}$ ,  $w \in R^{p_3}$ . In this study,  $u$  represents the input vector such as heat gains for thermal modelling and pollutant production for mass modelling.  $v$  and  $w$  are control vectors only available in thermal modelling, representing the HVAC inputs and heating/cooling setpoints, respectively.  $y$  is the output vector, representing the heating and cooling loads for thermal modelling,  $y \in R^q$ .  $A$  is the state (or system) matrix with a dimension of  $n \times n$ .  $B$ ,  $E$ , and  $F$  are the input matrices with a dimension of  $n \times p_1$ ,  $n \times p_2$ , and  $n \times p_3$  respectively.  $C$  is the output matrix with a dimension of  $q \times n$ .  $D$ ,  $G$ , and  $H$  are the feedthrough (or feedforward) matrices with a dimension of  $q \times p_1$ ,  $q \times p_2$ , and  $q \times p_3$  respectively. For thermal modelling in this study,  $G$  is a zero matrix.  $\dot{x}$  is the time derivative of the state variable.

$$\dot{x} = A \cdot x + B \cdot u + E \cdot v + F \cdot w \quad (2-72)$$

$$y = C \cdot x + D \cdot u + G \cdot v + H \cdot w \quad (2-73)$$

For thermal models, the state variables in  $x$  include surface temperatures of an envelope (interior and exterior) and the space temperature. Therefore, for a building case with  $f$  surfaces and  $s$  spaces, the dimension of  $x$  for annual hourly simulation (8760 hours annually) is  $(2f+s) \times 8760$ .  $u$  contains thermal boundary conditions (ambient air temperature and ground temperature) and heat gains on each surface and space (including solar heat gain, longwave radiative heat gain, and other internal heat gains such as infiltrated, lighting, occupant, equipment heat gains). So the dimension of  $u$  is  $(2+4f+4s) \times 8760$ .  $w$  is the heating or cooling energy input in the space with a dimension of  $s \times 8760$ .  $v$  is the heating and cooling setpoint (or setback) temperature of the space and its dimension is  $2s \times 8760$ . For mass modelling, the state variables in  $x$  represents the concentration of a specific pollutant in the space, with a dimension of  $s \times 8760$ .  $u$  includes the pollutant boundary condition (ambient pollutant concentration) and pollutant removals and generations in each space (internal pollutant sources and sinks). The dimension of the input matrix  $u$  is  $(2s+1) \times 8760$ .

#### 2.2.4.2 Model discretization

The SSM models are discretized to transfer continuous functions, models, variables, and equations into discrete counterparts. The discretization of the SSM models in this study is performed using Eqs. 2-74 to 2-81. The matrices with subscript of  $d$  are the discretized forms, where  $t_s$  is the sampling time.

$$A_d = e^{A \cdot t_s} \quad (2-74)$$

$$B_d = A^{-1}(A_d - I)B \quad (2-75)$$

$$E_d = A^{-1}(A_d - I)E \quad (2-76)$$

$$F_d = A^{-1}(A_d - I)F \quad (2-77)$$

$$C_d = C \quad (2-78)$$

$$D_d = D \quad (2-79)$$

$$G_d = G \quad (2-80)$$

$$H_d = H \quad (2-81)$$

### 2.2.4.3 Model solving

Thermal and mass state variables are solved through the iteration of the SSM models established. Closed-loop simulations will be performed considering the operation of HVAC systems to address heating/cooling demand by the occupants indoors. The operation of HVAC systems is determined by the setpoint and setback temperatures. The schedule depends on either preset operation schedules or the occupancy of the room (occupancy-based control) or other control strategies. The

operations of other building modules (such as air cleaners and shading system) also depend on the control/operation strategies user set up, either fixed schedule, occupancy-based schedule, or other control strategies (e.g. CO<sub>2</sub>-level-based control or solar-radiation-based control). For building modules with variant schedules, the operation of the module is determined within each time-step iteration. Besides, for calculations considering the longwave radiant heat transfer between surfaces, during each time-step iteration, the thermal transfer between surfaces is calculated and applied as the heat gain/loss at the next time step. As the weather data and ambient pollutant data is usually collected hourly. In this study, the iteration is performed hourly. In this study, annual simulation is performed.

### **2.2.5 Optimization**

To optimize the application of various building technologies/systems in green building design, performance simulation models can also collaborate with optimization algorithms to determine their optimal configurations (Kheiri 2018; Nguyen et al. 2014). Multi-objective optimization (MOO) is widely used to optimize the building features to achieve the trade-off between different performance criteria, e.g. energy efficiency, IAQ, thermal comfort, and cost (Chegari et al. 2021; Diakaki et al. 2008; Ghaderian and Veysi 2021; W. Li et al. 2021; Hongbin Liu et al. 2013; Shaikh et al. 2018; N. Wang et al. 2014; X. Wei et al. 2015; B. Wu et al. 2021). MOO refers to finding the optimal solution values of more than one desired goal. The motivation for using MOO is because it does not require complicated equations, which consequently simplifies the problem. MOO has emerged as the preferable approach to tackling sustainability problems. The process of optimization is to minimize the objective cost functions (e.g. energy consumption and infection risk). The solution of MOO models is generally expressed as a set of Pareto optima, representing

optimal trade-offs between given criteria. In this study, a multi-objective genetic algorithm (MOGA) is applied, which is based on the process of the genetic algorithm; the population-based property of the genetic algorithm is well applied in MOGAs (Long et al. 2015).

### **2.3 Modeling and simulation platform (A plugin in Rhino Grasshopper)**

The algorithms and models of performance simulation and optimization are developed and tested by Matlab or Python (Shen et al. 2020; Shen and Zhang 2019). To better demonstrate the green building design process using the proposed module-based GDS platform, a graphic user interface (GUI) is built as a plugin in Rhino Grasshopper, which is a widely used programmable building design platform as introduced earlier in this paper. Grasshopper is a parametric and algorithmic modelling tool created by David Rutten at Robert McNeel & Associates (Robert McNeel and Associates 2009), which works with Rhino to allow a powerful and efficient new way of designing. It allows different types of algorithms in design including numeric, textual, audio-visual and haptic applications. Currently, numerous developers have contributed to the development of a variety of applications in Grasshopper, such as parametric geometry modelling, optimization algorithms, machine learning methods, and environmental calculations (Robert McNeel & Associates 2022a). As reviewed in Chapter 1, the BPS tools developed in Grasshopper include Ladybug, Honeybee and ArchSim that estimate energy use and/or solar radiation based on EnergyPlus and/or Radiance, CFD-based building or urban airflow simulation tools such as Butterfly, Eddy, and RhinoCFD, and urban energy simulation and optimization tools like Dragonfly (based on UWG and CitySim) (Robert McNeel & Associates 2022a). Rhino and Grasshopper provide powerful software development kits (SDKs) for developers, which allow developers to program applications through C#, Python or VB.

RhinoCommon is the cross-platform .NET plugin SDK for Rhino development, while Grasshopper SDK is the SDK for Grasshopper development (Robert McNeel & Associates 2022b; Robert McNeel and Associates 2009). Figure 2-15 shows the hierarchy of RhinoCommon API and Grasshopper API. Except the classes shown in Figure 2-15, functions in System and MathNet.Numerics packages are also used in this study that deal with systematic issues and numeric calculations as shown in Figure 2-6. The general logic of the development of a Grasshopper plugin is taking the geometric models created in Rhino as inputs, connecting the components developed in Grasshopper with the geometries from Rhino, performing algorithms/models within each component of Grasshopper, and exporting and/or visualizing modelling results. Therefore, the programming of the plugin in this study is roughly divided into three categories, including the functions/classes defining the components in Grasshopper (GH Components, i.e., representation of GDS modules), the functions/classes used for calculations through different algorithms (GDS Functions), and the functions/classes for developing the panels in Rhino (GDS Panels) for information inputs as illustrated in the schematic in Figure 2-16.

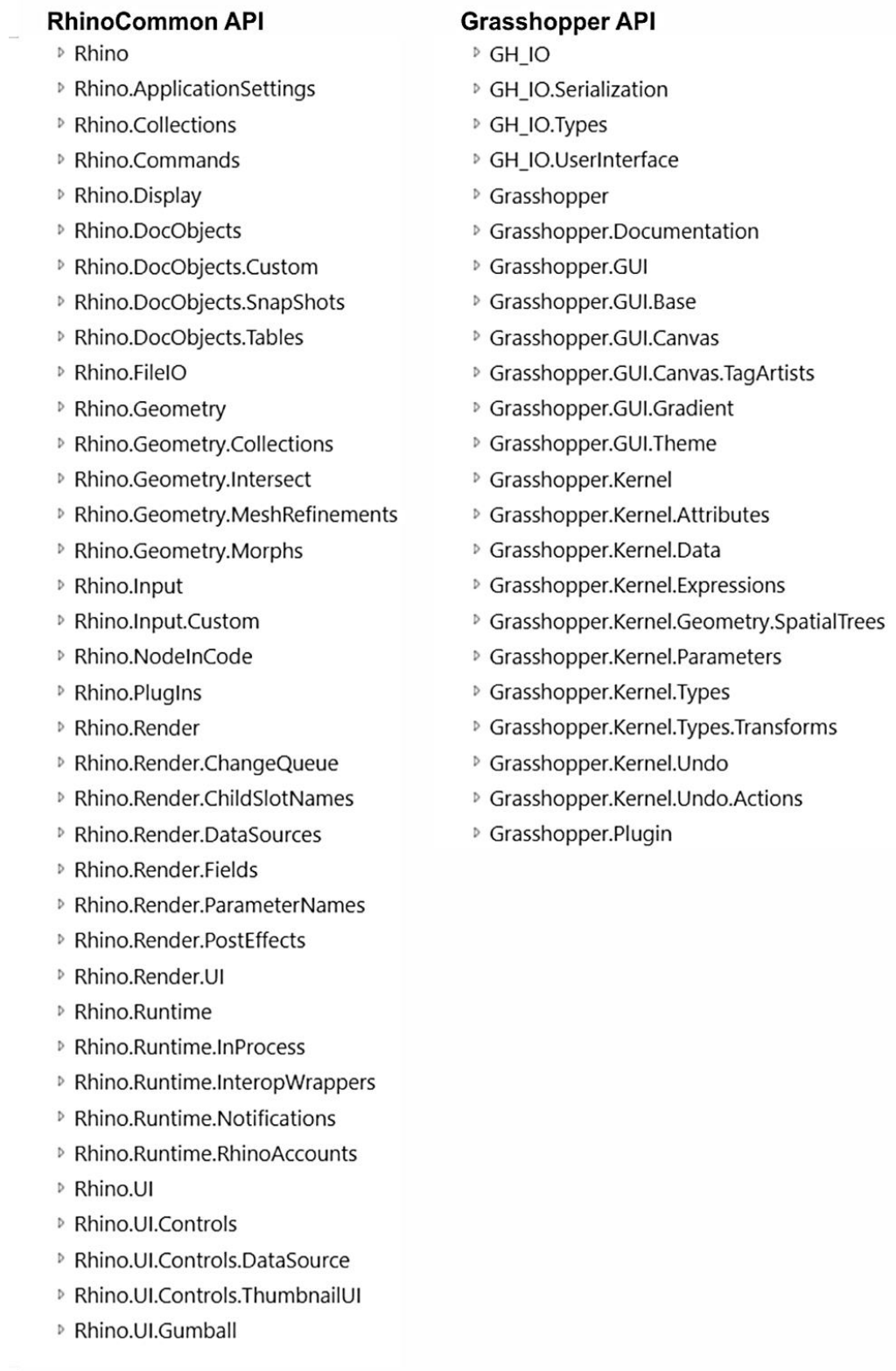


Figure 2-15. Structure of RhinoCommon API and Grasshopper API.



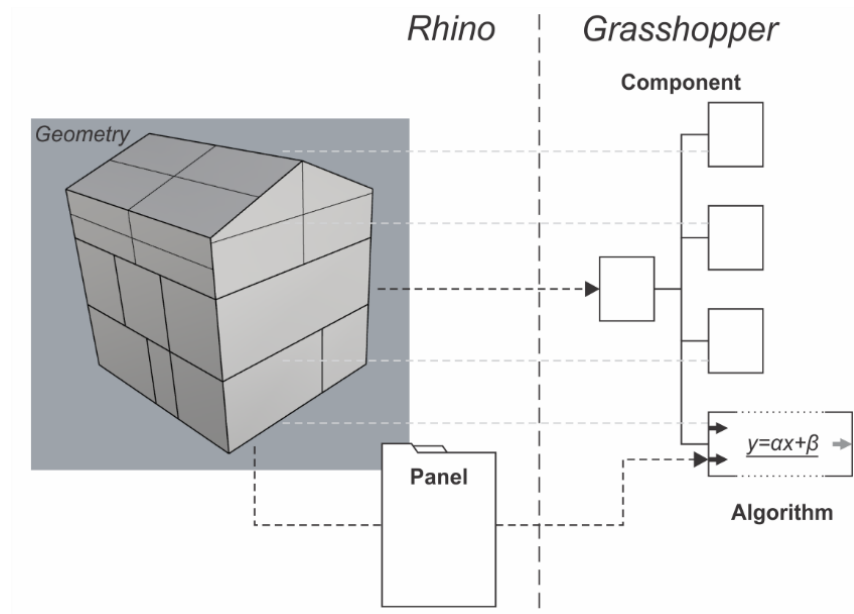


Figure 2-16. Schematic of workflow of Grasshopper components with Rhino geometries.

The components in Grasshopper are inherited from the `GH_Component` class from `Grasshopper.Kernal`. The component needs to be initialized first to define its name, description, category, and the father plugin's name. In this study, all components are registered under the `GreenDesignStudio` plugin. Then developers should register all input parameters and output parameters, with a variety of categories such as geometry (e.g. point, surface, box, etc.), number, Boolean, matrix, mesh, text parameters, etc. The geometry inputs are usually collected from the selections in Rhino UI, and the other inputs can be the outputs from other components or manual inputs by users. The structure of input and output parameters can be an item, a list, or a tree. The method of the component is defined in the `SolveInstance` property to perform and solve calculation using the data collected from the inputs and export the results to outputs. It is also necessary to

provide an icon for the component. The Exposure property controls where in the panel a component icon will appear. Besides the mandatory properties, developers can also override properties such as BeforeSolveInstance or define their own properties to add advanced functions before or after the method of the component.

The development of general algorithms has less restrictions than the development of components. Properties such as classes and functions can be defined for backend calculations. They are used for advanced processing or calculations that work with components. The development of panels in Rhino is based on the Rhino.PlugIns.PlugIn property in RhinoCommon. Commands can also be defined to perform certain commands by certain operations in Rhino. The development of panels is optional in this study. The main purpose of them is to provide a more straightforward approach to define some features to the geometry created in Rhino. In this study, building and room functional types can be defined through the GreenDesignStudio panel, e.g., bedroom, living room, and kitchen for residential buildings, conference room, open plan office, and lobby for offices. This information can also be assigned in Grasshopper through the developed components. But it is more straightforward to be defined through the panel in Rhino. More panels that can assign inputs or change parameters can be developed in future. But in this study, the development of panel is just for demonstration and is only used for representing building/room types. The hierarchy of the properties developed in this study is shown in Table 2-21. A brief description of each function/class is also presented.

Table 2-21. Functions and classes defined in the GreenDesignStudio plugin.

Category	Property	Description
GH Components  (GDS modules)	GHCommunity	Component that defines a community module (a bunch of building modules) from the selection of geometries in Rhino
	GHBuilding	Component that defines a building module from the selection of geometries in Rhino
	GHBuildingInfo	Component that can edit/update detailed building information
	GHSpace	Component that defines a space module
	GHEnvelope	Component that defines an envelope module
	GHEnvelopeInfo	Component that can edit/update detailed envelope information
	GHConstruction	Component that defines a construction
	GHMaterial	Component that defines a material
	GHOccupancy	Component that defines an occupancy module

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GHSchedule	Component that defines and contains schedules
GHHVAC	Component that defines a HVAC system with certain settings (e.g., airflow rate, set point temperature, etc.)
GHLight	Component that defines a lighting system
GHEquipment	Component that defines an equipment
GHAirCleaner	Component that defines an air cleaner
GHRenewableEnergy	Component that defines a renewable energy generation system
GHGreenModules	Component that contains green modules
GHDBConstruction	Component that contains datasets of construction in the systematical database
GHDBMaterial	Component that contains datasets of materials in the systematical database
GHFFD	Component that performs fast fluid dynamics (FFD) (not completed)

	GHSimulation	Component that performs BPS
	GHResults	Component that processes simulation results
	GHComparison	Component that compares simulation results from multiple cases
GDS Functions	GDSModules	Functions/classes that define building modules
	GDSSimulation	Functions/classes that perform BPS
	GDSGreenModules	Functions/classes that contain green modules
	GDSReference	Definitions of data structures for key classes/variables
	GDSExtensions	Library of factors, materials, schedules, etc.
GDS Panels	GDSPanelPlugIn	Panel instance
	GDSPanelCommand	Commands to the actions from the panel

Figure 2-17 shows the detailed inputs and outputs for major GH components of the GDS plug-in, and Figure 2-18 illustrates the rational of connections between different components of a simple example. Building component creates the GDS\_Building instance that contains detailed settings for building, zone, envelope, construction, occupancy, HVAC, lighting, equipment, and other facilities. The default settings are generated based on the reference settings for a specific type of building/space from standards or practices. The GDS\_Building instance is generated from the geometries (Boundary REPresentation, or BREP) created in Rhino. Each BREP block is created to represent a single space/zone/room. The combination of multiple BREP blocks forms a building model. The connection between different BREP blocks is processed by determining whether two adjacent blocks are intersected by the same surface. The surface is marked as an interior surface, and set up by specific characteristics for a certain type of interior surface (e.g. interior wall, ceiling, floor, or interior window).

The generated GDS\_Building instance then exports to the building information component to change its settings such as HVAC system, envelope settings, lighting, and equipment settings, if necessary. Envelope information components can be used to change the settings of a specific type of envelope of the building. For that selected type of envelope, the construction, infiltration coefficient, and window and shading settings will be set up. Special features of the envelope can also be added, e.g., solar chimney. HVAC components are created to change the HVAC settings for the whole building. Users can set up the HVAC variables like mode (e.g., ideal system), airflow rate, recirculated fraction, filter efficiency, other in-duct air cleaning efficiency, setpoint/setback temperature control and settings, and air delivery method (e.g., mixing ventilation, displacement ventilation), and occupancy sensing feature. Lighting and equipment components can set up the

power, working schedule, control method, and pollutant emission rate. Air cleaner components are used to set up the power, pollutant removal efficiency, secondary emission (if it exists), and working schedule and control method. Renewable energy generation component can change the settings for a specific type of renewable energy generation technology. For example, for a PV panel, the conversion coefficient and delivery coefficient of the energy generation, PV panel area, activated fraction of the area, and PV panel direction (Azimuth and Zenith angles), and the control method, can be set up. Simulation component imports the generated and update GDS\_Building instance as the input, and run the simulation with certain weather and air quality conditions collected and parsed from the EPW file (for thermal modelling) and the data provided by U.S. EPA (AQS API (U.S. EPA 2023)). Result component can import the simulation results, generate export files containing state variables (surface and air temperatures, pollutant concentrations), and visualize the result in Rhino Grasshopper. The result data visualization can also be performed by incorporating the components in other plug-ins, e.g., the data visualization components in Ladybug. The performance of a certain green building technology can be presented by comparing the case with proposed green technologies and the reference without green features.

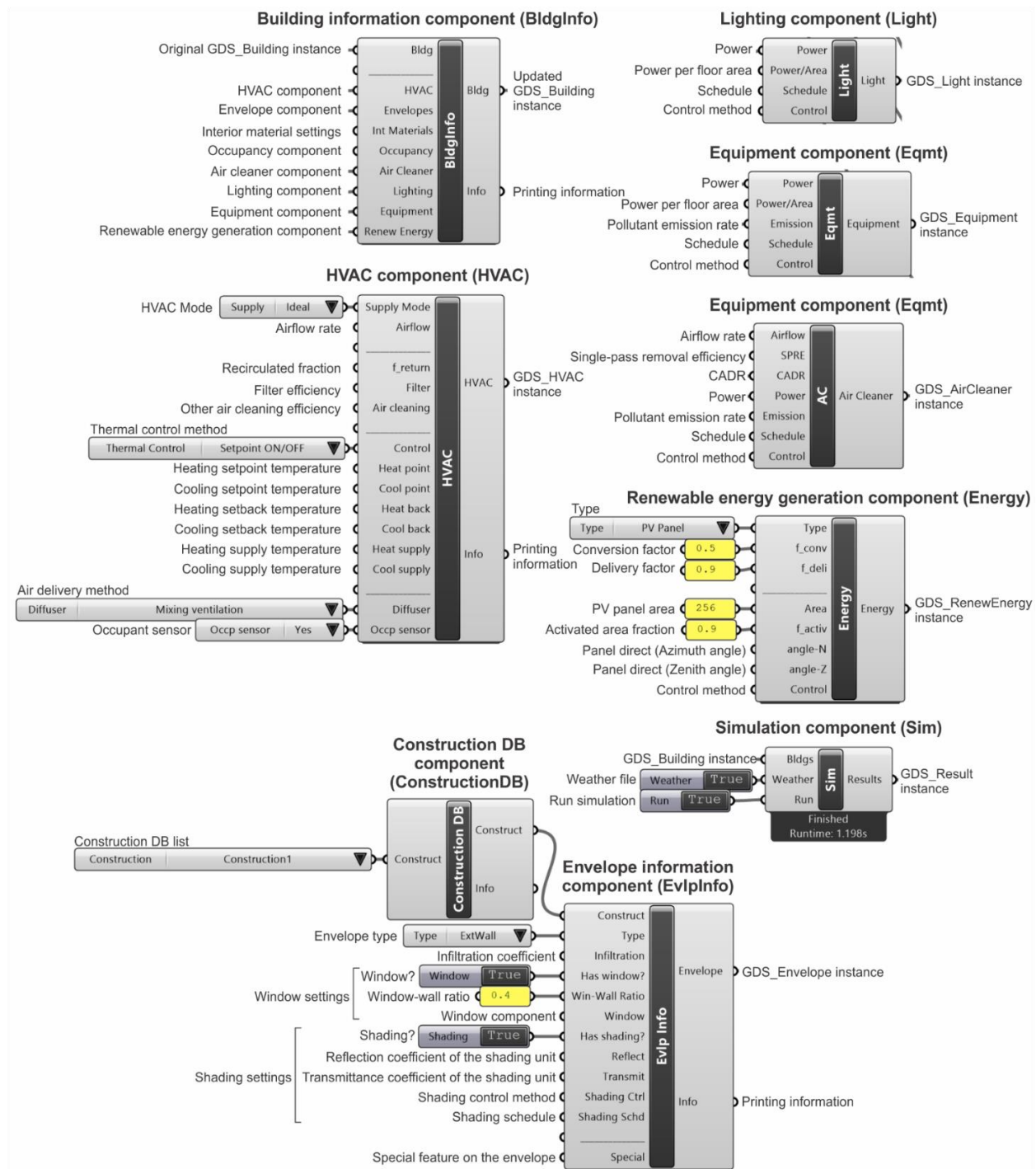


Figure 2-17. Major GH compounds (GDS modules) of the GDS plug-in.



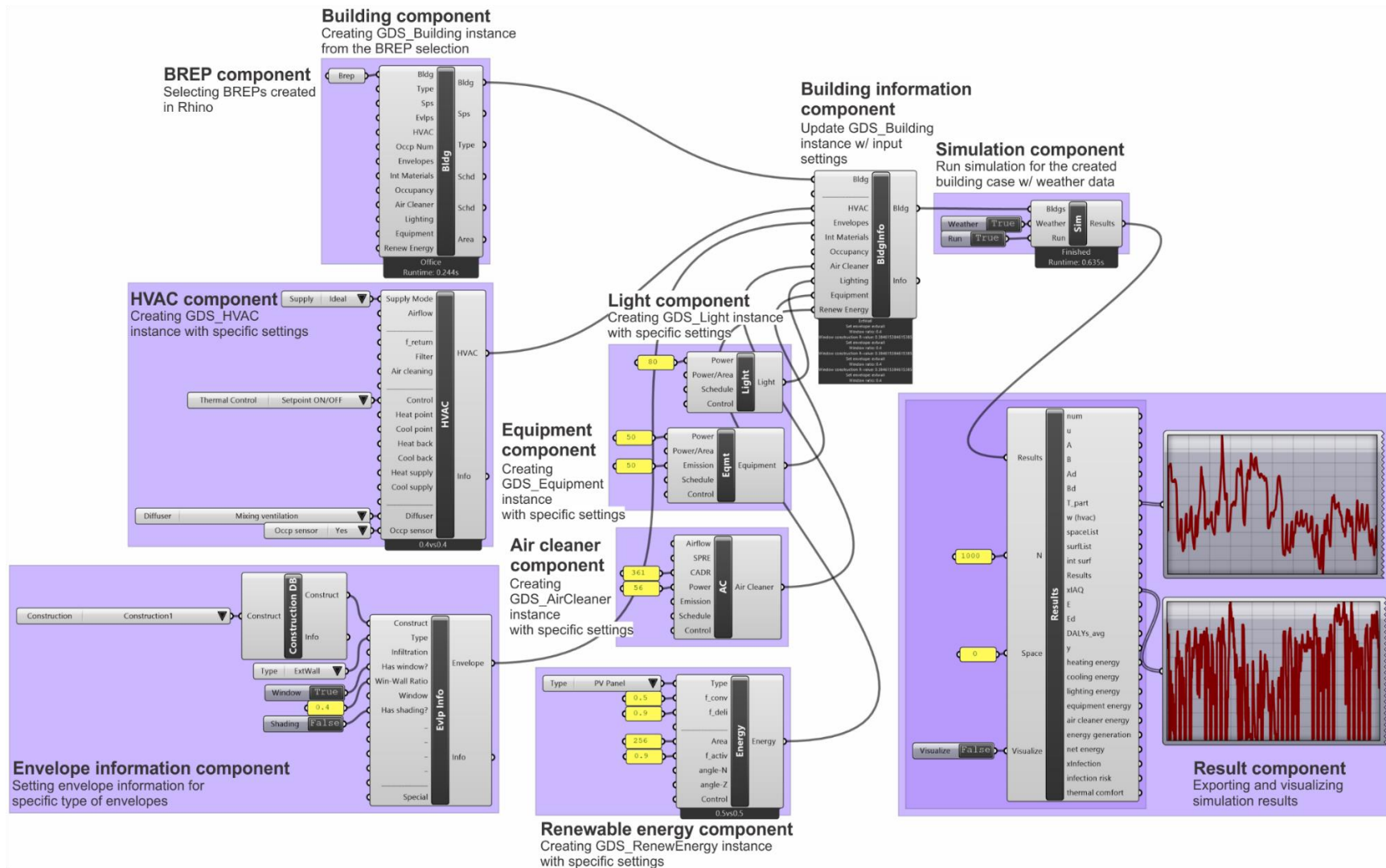


Figure 2-18. An example of the application of GDS plugin.

## 2.4 Case study

Case study is conducted to demonstrate the application of the developed GDS plug-in for building performance analysis and green building design. Two study cases are presented, including a DOE prototype building (small office) and the Syracuse COE headquarter building.

### 2.4.1 DOE prototype building - Small office case

#### 2.4.1.1 Case description

DOE prototype buildings were created by U.S. DOE, which cover 75% of the commercial building floor area in the U.S. for new construction, including both commercial buildings and mid- to high-rise residential buildings, and across all U.S. climate zones (U.S. DOE 2013). The case files were generated for EnergyPlus modelling (IDF files), comply with the ASHRAE 90.1 and IECC standard. The small office prototype building is a one-floor building with an unconditioned attic underneath the pitch roof (Figure 2-19). The building has four perimeter zones and one central zone used for office scenario. The room settings are shown in Table 2-22.

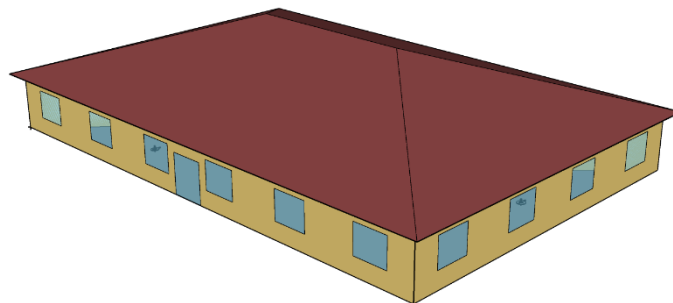


Figure 2-19. DOE prototype building of small office (U.S. DOE 2013).

The building has a total of 510 m<sup>2</sup> of conditioning area. The window-wall ratio on exterior walls was set as 0.4. Each window has a U-value of U-0.45 and a SHGC of 0.38. Exterior walls with R-20, interior walls with R-5, floor with R-15, ceiling with R-45, and exterior roof with R-30 were set up in the simulation. Envelope thermal properties were derived from the IDF file of the original prototype case and align with the ASHRAE 90.1 Standard. The maximum air infiltration on each exterior surface is 1.8 m<sup>3</sup>/h·m<sup>2</sup>, while during the simulation operation a 0.25 ratio of the maximum infiltration was applied on the surface. Lighting and equipment were operated with the typical office schedules (Figure 2-9). The heat generation from lighting and equipment aligned with the EnergyPlus setting. The occupant density defined in ASHRAE 62.1 Standard was used to estimate the occupancy number in offices (0.05 person/m<sup>2</sup>), resulting in 29 occupants in total. The occupancy schedules in Figure 2-8 were used. Occupants were seated or walking with 1 MET activity level when presented in the building. Heat generation from occupancy activities was 104 W with 40% convective heat that considered by the simulation. The clothing condition by occupants was assumed to be 1 clo in winter and 0.5 clo in other seasons.

The study case was simulated with the climate in Syracuse, NY. EnergyPlus weather (EPW) file of Syracuse was applied to the case. Building energy consumption, thermal comfort, IAQ (4 typical indoor contaminants, i.e. PM<sub>2.5</sub>, ozone, CO<sub>2</sub>, formaldehyde), health impacts and infection risks of COVID-19 were simulated. Ambient PM<sub>2.5</sub> and ozone concentrations were collected from EPA Air Quality System (AQS) database. Ambient CO<sub>2</sub> level was assumed to be 400 ppm while outdoor formaldehyde was assumed to be 7.2 µg/m<sup>3</sup>, which was suggested by the WHO report (WHO 2010). Data collected from Brazil, Canada, Germany, Italy, Mexico, Netherlands, and U.S., provided ambient concentrations of 1.5-16.4 µg/m<sup>3</sup> with a mean value of 7.2 µg/m<sup>3</sup> (SD = 5.1

$\mu\text{g}/\text{m}^3$ ). Outdoor concentration variations of ambient CO<sub>2</sub> and formaldehyde were not considered in this study. Indoor pollutant emission sources included occupant activities and building materials, furniture, and decorations. Each occupant was assumed to generate PM<sub>2.5</sub> in 10 mg/h, 13.9 L/h CO<sub>2</sub> (based on estimation using Eq. 2-34 (M. Li et al. 2022; Persily and de Jonge 2017)). Indoor materials generate 500  $\mu\text{g}/\text{h}$  formaldehyde in each zone. Natural decay of PM<sub>2.5</sub> in indoor environments was assumed to be 0.25 h<sup>-1</sup> (Shen, Kong, Dong, et al. 2021a) and natural decay of ozone was 4 h<sup>-1</sup> (Yao and Zhao 2018). Infection risks were simulated with the assumption of one infector was presented in each zone during the occupied period of the zone. Each infector produced 100 quanta/h virus-laden aerosols to the space (Shen, Kong, Dong, et al. 2021a, 2021b). Natural decay of virus-laden aerosols included particle natural deposition of 0.25 h<sup>-1</sup> and natural virus inactivation of 0.63 h<sup>-1</sup> in the air (Shen, Kong, Dong, et al. 2021a, 2021b).

An ideal HVAC system was used to provide conditioned air to meet the heating and cooling requirements of the building. Air volume was variable depending on the requirement. A simple ON/OFF control was applied to control the operation of the system. When the indoor air meets the heating/cooling setpoint/setback temperature, the system was off. Otherwise, the system would be operated to provide conditioned air to the space until the temperature meets the criteria again. The system was assumed to provide 25% outdoor air as the reference. A MERV 8 filter was used to filtrate the intake air.

Table 2-22. Simulation settings.

Zones	Type	Zone south (ZS): Office
		Zone north (ZN): Office
		Zone east (ZE): Office
		Zone west (ZW): Office
		Zone central (ZC): Office
		Attic: Unconditioned zone
Geometry	Dimensions	Zone south (ZS): $113 \text{ m}^2 \times 3.05 \text{ m} = 345 \text{ m}^3$
		Zone north (ZN): $113 \text{ m}^2 \times 3.05 \text{ m} = 345 \text{ m}^3$
		Zone east (ZE): $67 \text{ m}^2 \times 3.05 \text{ m} = 204 \text{ m}^3$
		Zone west (ZW): $67 \text{ m}^2 \times 3.05 \text{ m} = 204 \text{ m}^3$
		Zone central (ZC): $150 \text{ m}^2 \times 3.05 \text{ m} = 458 \text{ m}^3$
		Attic: $568 \text{ m}^2$ with $3.28 \text{ m}$ height, $720 \text{ m}^3$ in volume (pitched roof)
Envelope	Window setting	Window-wall ratio on exterior walls: 0.4
		U-value: U-0.45
		SHGC: 0.38
	Surface thermal properties	Exterior wall: $R-20$ , $C_p = 1090 \text{ J/kg}\cdot\text{K}$ , $\rho = 800 \text{ kg/m}^3$ , $\theta = 0.1 \text{ m}$
		Interior wall: $R-5$ , $C_p = 1090 \text{ J/kg}\cdot\text{K}$ , $\rho = 800 \text{ kg/m}^3$ , $\theta = 0.05 \text{ m}$
		Floor: $R-15$ , $C_p = 1500 \text{ J/kg}\cdot\text{K}$ , $\rho = 2000 \text{ kg/m}^3$ , $\theta = 0.2 \text{ m}$

Ceiling: R-45, $C_p = 1090 \text{ J/kg}\cdot\text{K}$ , $\rho = 800 \text{ kg/m}^3$ , $\theta = 0.1 \text{ m}$		
Exterior roof: R-30, $C_p = 1090 \text{ J/kg}\cdot\text{K}$ , $\rho = 800 \text{ kg/m}^3$ , $\theta = 0.1 \text{ m}$		
	Infiltration	$1.8 \text{ m}^3/\text{h}\cdot\text{m}^2$ maximum on exterior surfaces (schedule ratio: 0.25)
Lighting	Intensity	$6.89 \text{ W/m}^2$ (10% convective heat)
	Schedule	See Figure 2-9
Equipment	Intensity	$6.78 \text{ W/m}^2$ (convective heat)
	Schedule	See Figure 2-9
Occupancy	Density	$0.05 \text{ person/m}^2$ per ASHRAE 62.1 Standard (29 occupants in total)
	Schedule	See Figure 2-8.
	Activity	Seated when occupied (1 MET)
Skin area: $1.8 \text{ m}^2$		
Heat generation per person: 104 W (40% convective heat)		
Clothing: 1 clo in winter and 0.5 clo in other seasons		
Indoor pollutant emissions	Pollutant	PM2.5: $10 \text{ mg/h}\cdot\text{person}^a$
	generations	Ozone: 0 ( $5 \text{ }\mu\text{g/h}$ if an upper-room UVGI system used)
		CO2: $13.9 \text{ L/h}\cdot\text{person}$ (1 MET, BMR = 8) (M. Li et al. 2022;
		Persily and de Jonge 2017)

		Virus quanta: 100 quanta/h·infectors (1 infectors assumed in each zone)
		Formaldehyde: 500 µg/h from materials, furniture and decorations in each room
Indoor pollutant removal	Pollutant removals	PM2.5: 0.25 h <sup>-1</sup> (Shen, Kong, Dong, et al. 2021a)
		Ozone: 4 h <sup>-1</sup> (Yao and Zhao 2018)
		CO <sub>2</sub> : 0
		Virus quanta: 0.25 h <sup>-1</sup> (natural deposition) + 0.63 h <sup>-1</sup> (natural inactivation) (Shen, Kong, Dong, et al. 2021a, 2021b)
		Formaldehyde: 0
HVAC	Setpoint	Heating: 23 °C
		Cooling: 26 °C
		Setback
		Heating: 12 °C
		Cooling: 28 °C
		Supply temperature
		Heating: 38 °C
		Cooling: 13 °C
		Airflow rate
		Variable air volume to meet the thermal requirement

Outdoor air fraction	0.25
Filter efficiency	MERV 8 filter: 50% for PM2.5 and virus-laden aerosols, 0% for removing ozone, CO2 and formaldehyde.

<sup>a</sup> Walking-induced particle emission and resuspension was around 60 mg/h per room (Practice et al. 2016). Assuming 6 occupants in each room (similar to the proposed case), each occupant generated 10 mg/h particles.

#### **2.4.1.2 Building performance simulation**

Annual energy and IAQ performance of the reference office building were performed. The simulation was performed on an Intel Core i7-8750H CPU on a Windows laptop with a 16 GB memory. The simulation finished in 36.6 seconds. Figure 2-20 shows the indoor air temperature change over the simulated period in Zone South. Indoor air temperature can be maintained at either setpoint or setback temperatures very well. Heating and cooling energy accounts for the majority part of the total energy consumption by the building. Energy used by lighting and equipment systems also make a great contribution to the total energy use, while energy consumed by the fan system of the HVAC system is very limited.



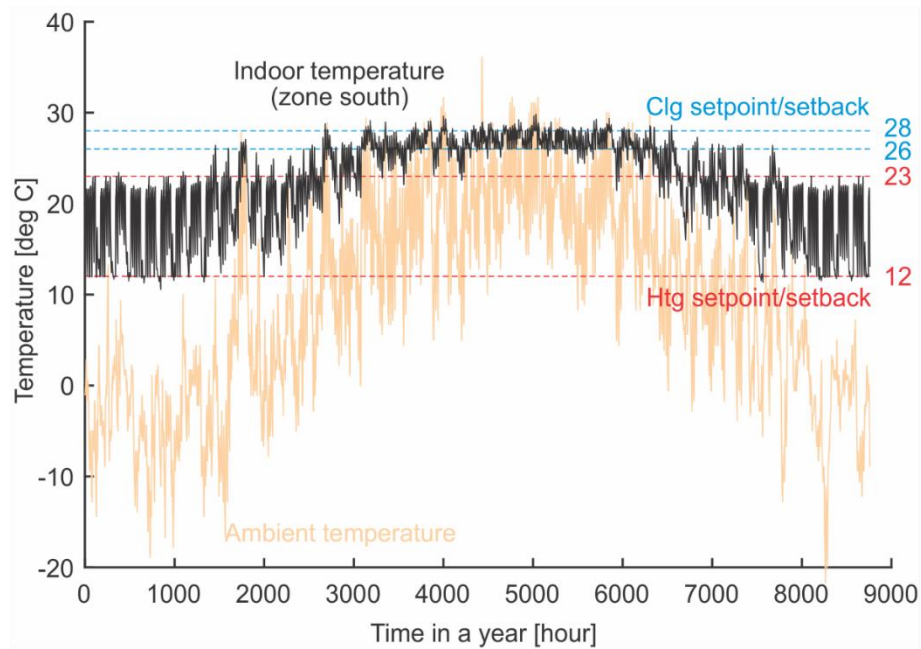


Figure 2-20. Indoor air temperature in Zone South of the simulated period.

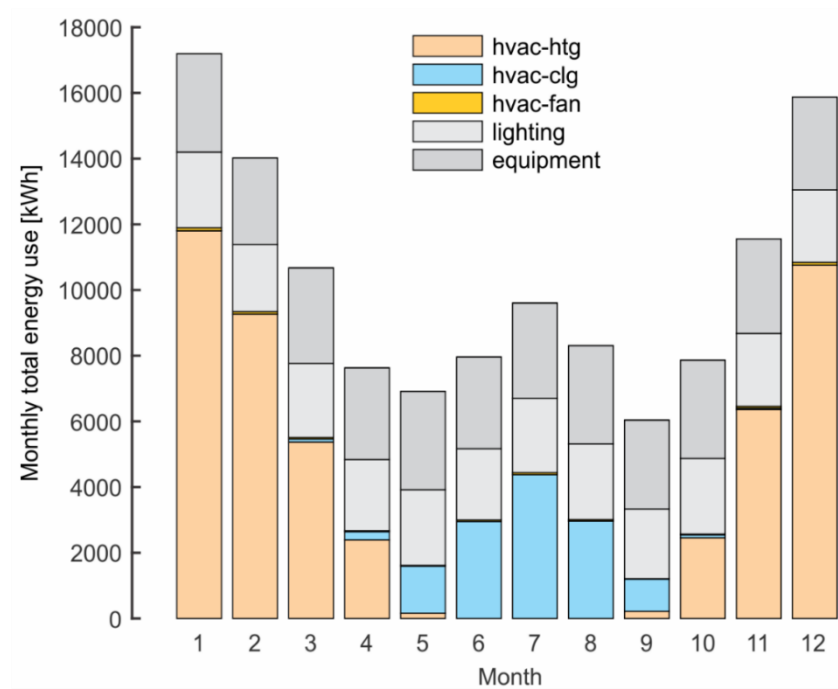


Figure 2-21. Monthly energy use by each category of the whole building.

Total energy use by the HVAC system (heating/cooling energy and fan power), lighting system, and equipment consumption is 123,636.4 kWh, with an average of 242.4 kWh per m<sup>2</sup> conditioned area (compared to 171.15 kWh/m<sup>2</sup> for offices from the EIA CBECS survey (U.S. EIA 2020b)). The average thermal comfort during the occupied period in each space is 66.1%. Average indoor concentrations of PM2.5, ozone, CO2 and formaldehyde are 4.0 mg/m<sup>3</sup>, 5.9 ppb, 585.6 ppm, and 9.1 µg/m<sup>3</sup>, respectively. Formaldehyde has the greatest health impact with average DALYs of 1089.6 years. PM2.5 caused DALYs of 39.8 years and ozone caused almost 0 years due to the lower indoor level. The average daily infection risk in the building is around 33.6% assuming an average exposure duration of 8 h a day. Therefore, the reference case has certain potential of energy saving and IAQ improvement to reach a better performance on IAQ and energy efficiency.

Table 2-23. Building performance of the small office prototype.

Energy use	Annual energy use [kWh]	123,636.4
	Annual energy use per conditioned area [kWh/m <sup>2</sup> ]	242.4
Thermal comfort	Average thermal comfort during occupied periods [%]	66.1
Pollutant concentrations	Average PM2.5 concentrations during occupied periods [mg/m <sup>3</sup> ]	4.0
	Average ozone concentrations during occupied periods [ppb]	5.9
	Average CO2 concentrations during occupied periods [ppm]	585.6

	Average formaldehyde concentrations during occupied periods [ $\mu\text{g}/\text{m}^3$ ]	9.1
Health impacts	DALYs due to PM2.5 exposure [yrs]	39.8
	DALYs due to ozone exposure [yrs]	0.0007
	DALYs due to formaldehyde exposure [yrs]	1089.6
Infection risks	Average virus quanta during occupied periods [quanta]	0.17
	Average daily infection probability due to exposure to virus quanta [%]	33.6

#### 2.4.1.3 Evaluation of potential performance of green building technology

To improve the energy and IAQ performance of the tested building, six categories of green building technologies were applied to the reference case, including advanced envelope technologies (higher insulation, thermal mass, high-reflective envelope, window technologies, and better airtightness), HVAC technologies (return air ratio, and higher-efficiency filters), portable air cleaners, upper-room UVGI system, green indoor materials with less emissions, and personal protective equipment (PPE). The proposed case with green technologies was analyzed through the GDS approach. The performance improvement (%) of each technology is presented in Table 2-24.

Higher insulation envelopes can improve the building energy efficiency, thermal comfort, and health impacts due to exposure to indoor pollutants, while impairing infection risk mitigation efficiency. However, the improvement is very limited, probably due to the fact that the reference case already used very high insulation materials. It makes thermal insulation not the critical factor influencing energy consumption. Health impacts were affected by changing thermal insulation of the building envelope because the proposed HVAC system provided variable air volume to meet the change of heating/cooling load. When the heating/cooling load changes, supply air flow rate by the HVAC system will be changed. Then the ventilation supplied by the HVAC system will be changed as well. Thermal mass and window technologies do not affect the building performance very significantly either. But air infiltration through exterior walls has considerable impacts on energy and IAQ performance. If lower infiltration envelopes were applied, health impacts due to indoor pollutants will be impaired due to the accumulation of indoor pollutants. Introducing more outdoor air is good for diluting indoor contaminants, but will cause higher energy consumption. Increasing outdoor air ratio of the HVAC system will increase the energy consumption to condition the outdoor air and may introduce outdoor pollutant PM<sub>2.5</sub> and ozone and affect health impacts, but it can help to mitigate indoor emissions including formaldehyde and virus quanta. Using HEPA filter also benefits indoor health effects and infection risk mitigation efficiency. Adding carbon filter that removes ozone and formaldehyde readily will benefit indoor health impacts.

Although portable air cleaners may increase building energy use, but the penalty can be neglected compared to its considerable improvement on health impacts and risk mitigation effectiveness. The upper-room UVGI system is highly effective for removing virus quanta and mitigate infection

risk. But it also requires elevated energy consumption. Green materials with less pollutant (i.e. formaldehyde in this study) emissions is very effective for health improvement. Wearing masks is very efficient for mitigating infection risks.

Table 2-24. Performance of green building technologies.

<b>Technology</b>		<b>Performance improvement (compared to the reference case) [%]</b>			
		<b>Thermal comfort</b>	<b>Energy efficiency</b>	<b>DALYs</b>	<b>Risk mitigation</b>
Envelope	R-25 exterior wall	0.14	0.10	0.07	-0.03
	R-30 exterior wall	0.26	0.15	0.17	-0.03
	R-35 exterior wall	0.31	0.22	0.23	-0.04
	R-40 exterior wall	0.35	0.27	0.32	-0.05
	Thermal mass for exterior wall (Cp = 2000 J/kg-K)	0.06	0.05	0.09	-0.04
	High-reflective envelope (reflectance = 0.7)	-0.03	0.09	-0.24	-0.03
	Low U-value window (U-0.3)	0.46	0.31	0.69	-0.03

	High SHGC window (SHGC = 0.82)	0.50	-1.36	4.44	0.37
	High infiltration exterior wall (9 m <sup>3</sup> /h·m <sup>2</sup> )	-2.81	-2.56	42.51	3.42
	Low infiltration exterior wall (0.18 m <sup>3</sup> /h·m <sup>2</sup> )	0.79	0.52	-464.61	-1.16
HVAC	Return fraction = 0.5	0	-5.41	-0.61	0.89
	Return fraction = 0.25	0	-10.96	-1.09	1.68
	Return fraction = 0	0	-16.74	-1.48	2.41
	HEPA filter	0	0	0.11	2.40
	Activated carbon filter (90% on ozone removal and 50% on formaldehyde removal)	0	0	6.86	0
Air cleaner	1 air cleaner with 361 m <sup>3</sup> /h CADR for particles; 56W power	0.30	-1.30	0.30	46.18
	2 air cleaners	0.51	-2.55	0.40	62.73
	1 air cleaner with 361 m <sup>3</sup> /h CADR for particles and	0.30	-1.30	34.02	46.18

	carbon filter for 90%				
	ozone and 50%				
	formaldehyde removal;				
	56W power				
Upper-room UVGI system	UVGI system for inactivating virus (12 h <sup>-1</sup> equivalent ACH provided during the occupied period)	0	-3.75	0	88.41
Green material	Less emission materials (50% less emission of formaldehyde)	0	0	30.05	0
PPE	Cloth masks (50% on particles)	0	0	0	68.68
	Surgical masks (75% on particles)	0	0	0	91.69
	N95 masks (95% on particles)	0	0	0	99.66

#### 2.4.1.4 Optimization of energy and IAQ strategies

The above analysis can provide the individual performance of each building module. However, it remains unknown how different building modules interact with others. During the pandemic, many IAQ improvement strategies have been proposed or implemented to minimize the infection risk. However, many of these strategies require considerable extra costs including energy use. Therefore, it is essential to find a tradeoff between risk mitigation and energy efficiency, especially considering longer term applications of these technologies. In this study, a MOO was performed to achieve the optimal design of building modules in terms of both energy efficiency and infection risk mitigation. A MOGA with a population size of 100 and a generation size of 20 was constructed to minimize energy use per area and infection risk. Parameters including envelope R-value, specific heat, absorption, infiltration coefficient, return air ratio, filter efficiency, air cleaner efficiency, material emissions, UVGI system, and air distribution.

Figure 2-22 shows the Pareto front of the optimization, indicating the optimal design features and objective outputs of the case. It can be found that under certain restraints, energy use of the building can be as low as  $171 \text{ kWh/m}^2$ , while infection probability can be below 0.017%. If we assume the energy use per area has to be below  $175 \text{ kWh/m}^2$ , the marked point at the Pareto front will be the optimal design of the building under the current settings. Then the parameter settings in Table 2-25 are the optimal settings for the current target. Compared to the reference case, the optimal design can save up to 27.8% energy use while mitigate more than 99% infection risk.



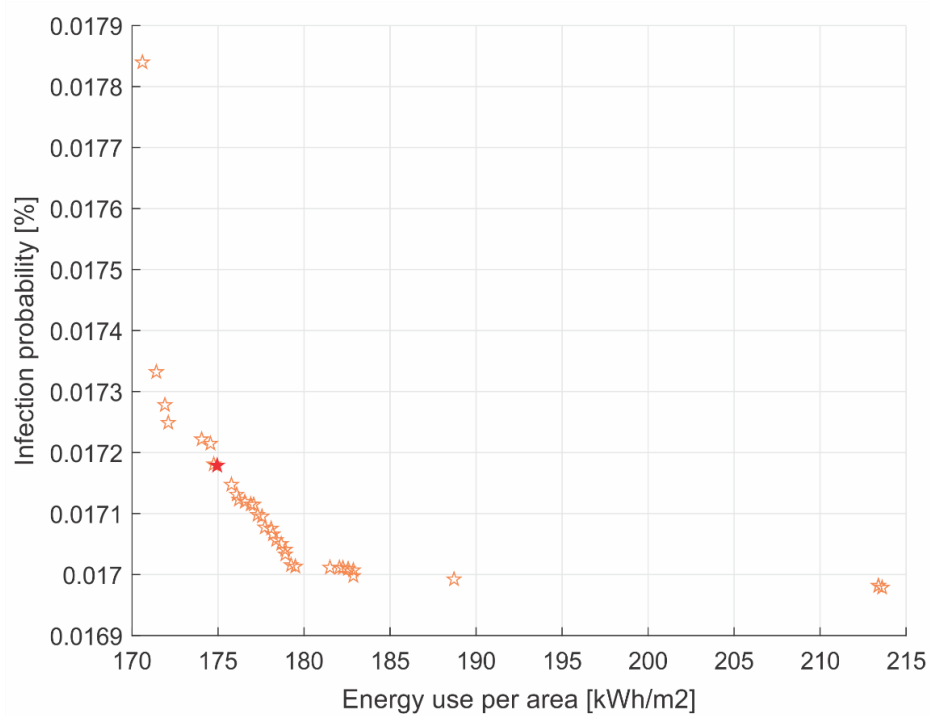


Figure 2-22. Pareto front of the optimization study.

Table 2-25. Optimal settings regarding energy efficiency and risk mitigation.

Parameter	Value
Exterior wall R-value	R-29
Exterior wall Cp [J/kgK]	1264
Exterior wall absorptance	0.39
Exterior roof R-value	R-26
Exterior roof Cp [J/kgK]	1433
Exterior roof absorptance	0.66

Window U-value	U-0.28
Window SHGC	0.69
Exterior wall infiltration coefficient	0.16
Return air ratio	0.71
Air distribution efficiency	2 (displacement ventilation)
Filter	HEPA
Air cleaner	HEPA, on
Air cleaner heat [W]	103
Air cleaner airflow rate [m <sup>3</sup> /h]	502
UVGI system	On
UVGI efficiency [h <sup>-1</sup> ]	9
PPE	N95 mask (95% removal)

#### 2.4.2 Energy and IAQ analysis in a section of Syracuse COE headquarter building

The Syracuse Center of Excellence (COE) headquarters office and laboratory building is utilized as the case study building, which is located in Syracuse, New York, US. The focus of the pilot study is on a section of the COE building, which includes a conference room (Room 203) and the affiliated south-facing corridor (Figure 2-23). The conference room is adjacent to the north side

and is connected to a part of a south-facing corridor. The south façade comprises full-height glazing and the interior walls separating the corridor and conference room utilize glass partitions for access to daylighting. The north façade is a steel-framed wall assembly with minimal glazing (window ratio around 0.4). The ceilings, floors and lateral walls/surfaces connect to adjacent rooms and corridor sections, and are therefore all interior partitions. In this study, the adiabatic boundary condition is adopted for these surfaces. Dimensions for the corridor and conference room are  $3\times 9\times 3\text{m}$  and  $15\times 9\times 3\text{ m}$  high, respectively (Figure 2-23).

According to the proposed modular method, the case study can be modularized following the structure in Figure 2-23, including the definition of zone modules, service system modules and enclosure modules. Some key module parameters of the study case are shown in Table 2-26. Five typical “green” strategies are applied in the building modules, including higher airtightness for building enclosure, super insulation for enclosure, thermal mass, controlled artificial lighting system and controlled shading system. The design parameters adopted for the reference case are defined based on the typical practices for conference rooms and corridors of an office building.

The airtightness of the reference case is 0.7 air changes per hour (ACH) for exterior windows. More airtight enclosures are applied to reduce heat loss or gain through infiltration in the target case, i.e. 0.5 ACH for exterior windows. Higher thermal insulation ( $R\text{-value} = 2.0\text{K}\cdot\text{m}^2/\text{W}$ ) is used for the exterior glazing surfaces on south and north façade compared to a reference  $R\text{-value}$  of  $1.0\text{K}\cdot\text{m}^2/\text{W}$ . Thermal mass is applied to the north wall with an elevated heat capacity of  $2000\text{J}/\text{kg}\cdot\text{K}$ . An intelligently controlled shading system is added to the south glazing façade of the

target case and it works when the solar irradiation on the south façade is higher than  $100\text{W/m}^2$  during summer (between June and August). The shading system is assumed to be able to reduce overall 70% of solar irradiation on the south façade, while the reference building does not have any shading system. The electric lighting schedule of the reference case follows the occupant schedule, which means the light is on when the corridor/room is occupied. The lighting intensity is 100lux ( $5\text{W/m}^2 \cdot 100\text{lux}$ ) for the corridor and 400lux ( $5\text{W/m}^2 \cdot 100\text{lux}$ ) for the conference room. In the target case, the lighting system is controlled depending on the daylighting intensity as well. The lighting system works following the same schedule as the reference case, but it will stop working when the daylighting can provide enough indoor illumination (100lux for corridor and 400lux for conference room).

To compare the performance of the proposed RC models in GDS and the conventional simulation model, an EnergyPlus model was developed to perform the energy and thermal simulation. Same settings of the RC models as introduced above were applied to the EnergyPlus model. Both GDS and EnergyPlus simulations were performed on the same computer (a Windows laptop with the Intel Core i7-8750H CPU and a 16 GB memory).

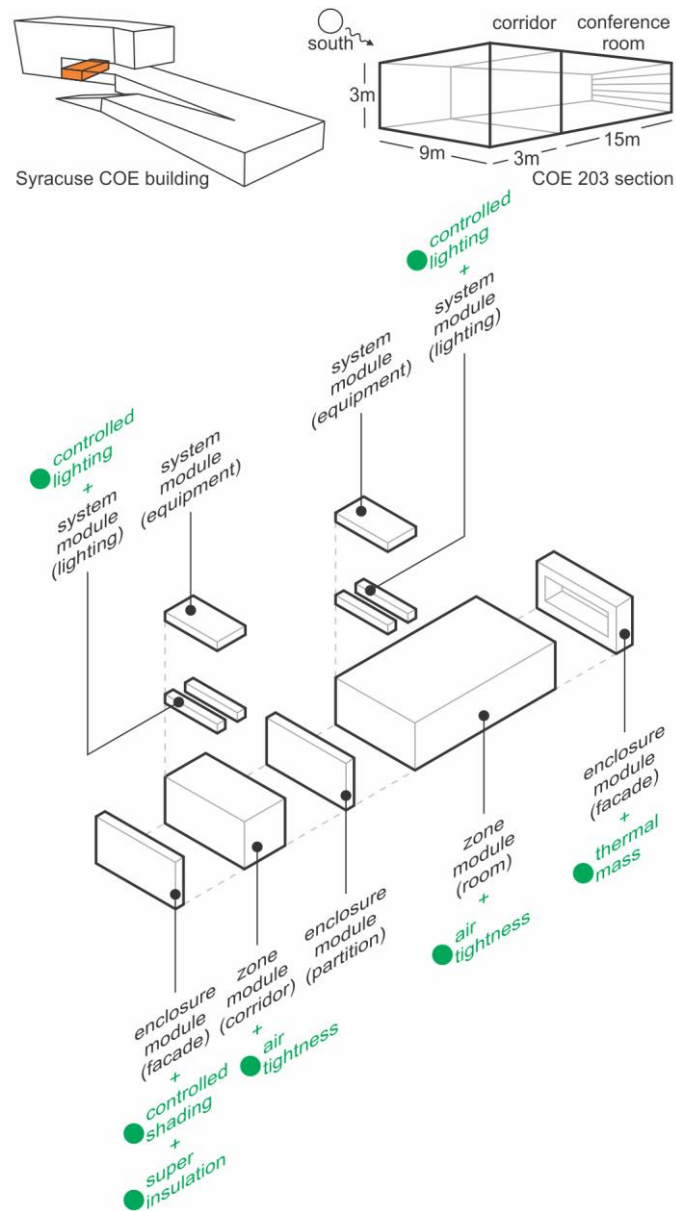


Figure 2-23. Schematic of the study case, COE building Room 203 section.

Table 2-26. Key parameters of building modules for the study case.

Module type	Name	Module parameters	Values	
			Target design	Reference design <sup>i</sup>
Zone	Corridor	Occupancy <sup>a</sup>	0.117/m <sup>2</sup> ; 140W/person;	
		Set point <sup>a,b</sup>	clg: 26/28°C; htg: 23/12°C;	
		<i>R-value</i> (int. conv.) <sup>e</sup>	0.3K·m <sup>2</sup> /W	
		Air properties	1.225kg/m <sup>3</sup> ; 1005J/kg·K;	
Zone	Room	Occupancy <sup>a</sup>	0.111/m <sup>2</sup> ; 123W/person;	
		Set point <sup>a,b</sup>	clg: 25/28°C; htg: 24/12°C;	
		<i>R-value</i> (int. conv.) <sup>e</sup>	1.2K·m <sup>2</sup> /W	
		Air properties	1.225kg/m <sup>3</sup> ; 1005J/kg·K;	
Service system (HVAC)	CorridorHVAC	Fresh air rate	1.0ACH	

		Filtration rate	100%	
		Heating Efficiency	0.5	
		Cooling COP	3.0	
Service system (HVAC)	RoomHVAC	Fresh air rate	1.0ACH	
		Filtration rate	100%	
		Heating Efficiency	0.5	
		Cooling COP	3.0	
Service system (lighting)	CorridorLighting	Illuminance <sup>a</sup>	Controlled lighting <sup>c</sup>	100lux; 5W/m <sup>2</sup> ·100lux;
Service system (lighting)	RoomLighting	Illuminance <sup>a</sup>	Controlled lighting <sup>c</sup>	400lux; 5W/m <sup>2</sup> ·100lux;
Service system (equipment)	CorridorEquipment	Heat gain <sup>a</sup>	1.85W/m <sup>2</sup>	
Service system (equipment)	RoomEquipment	Heat gain <sup>a</sup>	11.77W/m <sup>2</sup>	

Enclosure	SouthGlazingFacade	<i>R-value</i>	2.0K·m <sup>2</sup> /W <sup>g</sup>	1.0K·m <sup>2</sup> /W
		<i>C<sub>p</sub></i>	840J/kg·K	
		Density	2500kg/m <sup>3</sup>	
		Thickness	0.005m	
		SHGC	0.819	
		Airtightness	0.5ACH <sup>f</sup>	0.7ACH
		Shading	Controlled shading <sup>d</sup>	No shading
		Pollutant removal <sup>i</sup>	0.0108h <sup>-1</sup>	
Enclosure	GlazingPartition	<i>R-value</i>	1.5K·m <sup>2</sup> /W	
		<i>C<sub>p</sub></i>	840J/kg·K	
		Density	2500kg/m <sup>3</sup>	
		Thickness	0.005m	



		SHGC	0.768	
		Airtightness	0	0
		Shading	No shading	
		Pollutant removal	0.0108h <sup>-1</sup>	
Enclosure	NorthFacade	<i>R-value</i>	22.8K·m <sup>2</sup> /W	
		<i>C<sub>p</sub></i>	2000J/kg·K <sup>h</sup>	1000J/kg·K
		Density	1400kg/m <sup>3</sup>	
		Thickness	0.2m	
		Thermal absorption	0.7	
		Airtightness	0ACH	
		Pollutant removal	0.1058h <sup>-1</sup>	
Enclosure	NorthWindow	<i>R-value</i>	2.0K·m <sup>2</sup> /W <sup>g</sup>	1.0K·m <sup>2</sup> /W

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$C_p$	840J/kg·K	
Density	2500kg/m <sup>3</sup>	
Thickness	0.005m	
SHGC	0.819	
Airtightness	0.5ACH <sup>f</sup>	0.7ACH
Shading	No shading	
Pollutant removal	0.0009h <sup>-1</sup>	

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<sup>a</sup> These occupant parameters depend on specific schedules.

<sup>b</sup> clg means cooling and htg means heating. 25/28°C presents the set point temperature is 25°C while the setback temperature is 28°C. The set point temperature works between 6am and 7pm, while the setback temperature works for the other hours.

<sup>c</sup> The lighting system works following the same schedule as the reference case, but it will stop working when the daylighting can provide enough indoor illumination (100lux for corridor and 400lux for conference room).

<sup>d</sup> The shading system for south glazing façade will work when the solar irradiation on the south façade is higher than  $100\text{W/m}^2$  during summer (between June and August). When the shading system applied, the solar heat gain on the south façade need to be multiplied by a factor of 0.3.

<sup>e</sup> Convective *R-value* between the interior surface and air.

<sup>f</sup> Higher airtightness of building enclosure.

<sup>g</sup> Super insulation for exterior glazing façade.

<sup>h</sup> Thermal mass.

<sup>i</sup> Same as target building unless indicated otherwise.

<sup>j</sup> The ozone removal rate on building surface is estimated from literature (Shen and Gao 2018).

Performance metrics of energy consumption, percent of people satisfied with IAQ, thermal comfort and lighting quality are evaluated. The predicted annual performance metrics of the reference and target building by the RC model are shown in Table 2-27. The heating and cooling energy consumption is predicted assuming that the heating efficiency is 0.5 while the cooling COP is three. The target building can save considerable energy using these “green” strategies compared to the reference. People in the target building are more satisfied with the indoor air quality with more than 80% satisfaction when the indoor ozone limit is 35ppb (WHO 2005). The percent of people satisfied on indoor lighting quality is predicted when the recommended illuminance range for corridor is 50 to 150lux and the range for conference room is 300 to 500lux (IESNA 2011). The indoor illuminance is assumed to be uniformly distributed and the unevenness is not considered. The calculated lighting satisfied percent is almost zero because the indoor illuminance is far away from the recommended range due to the high-illuminance daylight transmitted through extensive glazing surfaces on façade. A more efficient shading strategy for south glazing façade is supposed to be used.

To evaluate the performance of different “green” strategies applied in the target building so as to find its best design strategy, the performance improvement potentials to the reference case are calculated based on different metrics (Figure 2-24). The table in Figure 2-24 is color-mapped by the magnitude of performance improvement potentials. A red cell means a positive potential, while a blue cell represents a negative effect. The tested five green strategies perform variably on the study case. According to the result, elevated airtightness for building enclosure can save considerable heating energy by reducing the heat loss through infiltration. But it will consume more energy for cooling due to the trapped internal heat from occupants, lighting, and incident

solar irradiation. An overall improvement for total energy use (i.e. combined heating, cooling, and lighting energy) is observed by applying higher airtightness in building enclosure. It will also improve the PPS on thermal comfort and IAQ. Lower infiltration results in the reduced ozone infiltrated from the outdoor air. Like the elevated airtightness, the higher insulation for exterior windows can save heating energy but consume more energy for cooling because of the trapped internal heat. A slight improvement is observed for the total energy efficiency. Higher insulation can slightly improve indoor thermal comfort in the corridor as well, but does not really work for improving thermal comfort in the conference room (almost 0%). It is probably because of the limited area of glazing surface on the north façade. Thermal mass does not appear to significantly affect the energy use or IEQ of the building (less than 0.5%) since the thermal mass material (concrete in this study) contributes limited to the whole building envelope. The controlled shading system can decrease the cooling energy consumption by reducing the solar heat gain in summer, while it does not work for reducing heating energy use. It has a limited improvement for total energy use for the whole building. At the same time, the controlled shading strategy will inversely impact the indoor thermal comfort, particularly for the corridor. The controlled lighting strategy can save massive lighting energy use (around 70%) since the daylight can provide considerable lighting to meet the illuminance requirement. Due to the reduced heat released from the electric lighting system, the heating energy use will increase while the cooling energy will decrease. Overall, the controlled lighting strategy will improve the total energy efficiency. However, an overall inverse impact on thermal comfort is observed, particularly for the room, which is mainly affected by the reduced internal heat gain during winter.

When applying all five strategies in the target building, the combined effect shows it will considerably reduce energy use and improve IAQ. But thermal comfort will not be improved due to the insufficient internal heat gain caused by the controlled shading and lighting strategy. Besides, neither of these strategies will affect the indoor lighting quality, likely because the incident illuminance is much higher than the recommended level. Considering that the thermal mass does not offer major improvement to the building performance in this study, and the controlled shading strategy cannot significantly improve energy efficiency and will even deteriorate the indoor thermal comfort. These two strategies are not recommended to be applied in the studied case. The other three strategies, i.e. higher airtightness, super insulation and controlled lighting, can be integrated in the building to better improve the energy efficiency but avoid the deterioration in IEQ. It can save around 14% of total energy use and elevate over 3% of people satisfied with the thermal comfort in corridor but have a slight decrease in the thermal comfort in room. People satisfied with IAQ will be improved by more than 6%. Therefore, in order to achieve a better performance and save the building cost, higher airtightness, elevated insulation, and controlled lighting strategy are recommended to be used in the design.

Table 2-27. Annual performance metrics of the reference and target building simulated by the RC model.

Case	Space	Energy use <sup>a</sup>				IEQ		
		Heating	Cooling	Lighting	Total energy use	PPS <sub>IAQ</sub> <sup>b</sup>	PPS <sub>ITC</sub>	PPS <sub>ILQ</sub>
		[kWh]	[kWh]	[kWh]	[kWh]	[%]	[%]	[%]
Reference	Corridor	14427	985	591	16004	74.1	46.0	0
	Conference room	30603	2313	11826	44742	79.5	33.3	0
	Total	45030	3298	12417	60746	/	/	/
Target	Corridor	13626	385	58	14069	79.9	42.5	0
(with 5 strategies)	Conference room	32660	1125	3561	37346	84.4	32.2	0
	Total	46286	1510	3619	51415	/	/	/

<sup>a</sup> The total annual energy consumption is predicted assuming that the heating efficiency is 0.5 while the cooling COP is three.

<sup>b</sup> The indoor ozone concentration limit is assumed to be 35ppb based on the WHO baseline (WHO 2005).

<sup>c</sup> The illuminance limit for corridor is assumed to be in the range of 50 to 150lux, while the limit for conference room is between 300 and 500lux (IESNA 2011).

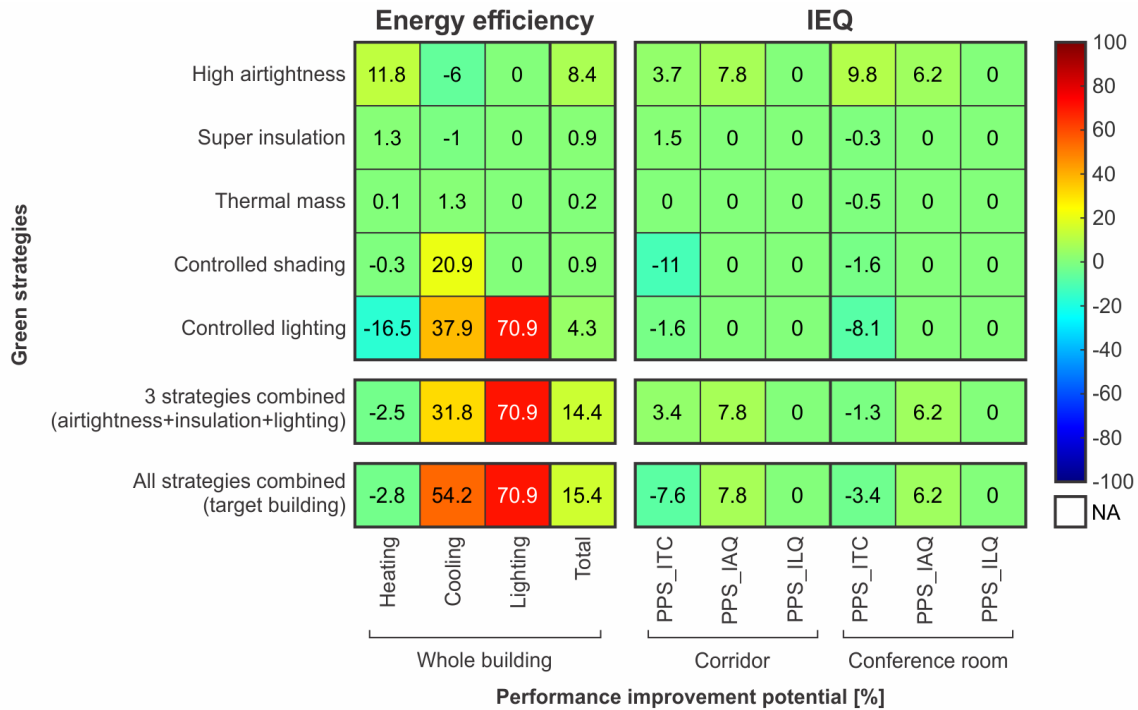


Figure 2-24. Performance improvement potentials of different green strategies.

The EnergyPlus simulation illustrates similar results. The comparison of predicted hourly air temperatures in the conference room and the corridor section of the reference case by RC model and EnergyPlus (EP) model are shown in Figure 2-25. The corresponding root mean squared errors (RMSE) of hourly air temperature between two models are 2.3 °C and 2.0 °C for the conference room and the corridor section, respectively. The runtime of the EnergyPlus model is 14.0 seconds while the runtime of the RC model is 3.6 seconds, which is significantly faster than the EnergyPlus model. The preparation process of the simulation is hard to quantify as it largely depends on users' experience and proficiency. But the application of GDS should be faster than the EnergyPlus simulation as the GDS model contains a lot of default settings, which can largely save time during the model establish process. However, the present example is a very simple case with only two



zones. When the model has a higher complexity (with more zones or surfaces), the simulation time using the RC model will increase accordingly.

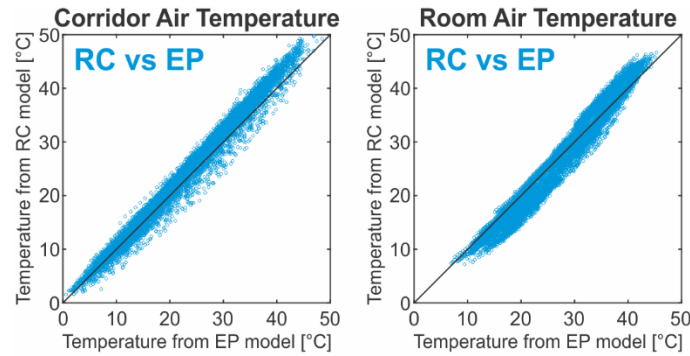


Figure 2-25. Comparison of predicted corridor and conference room air temperature from the RC model and EnergyPlus (EP) model.

### 2.4.3 Analysis and visualization of energy and thermal environment for the whole building of Syracuse COE headquarter

The whole building of Syracuse COE headquarter is also simulated to analyze its energy and IAQ performance. The Syracuse COE building is a 5-floor complex including both office and laboratory spaces (Figure 2-26). Figure 2-27 is a schematic of the 3D geometry model of the building in simulation. The building geometry model is simplified in certain content to simplify the simulation. The office space of the model has 56 zones across 5 floors, including open offices, small offices, conference rooms, lobbies, lounges, restrooms, dining spaces, and circulation spaces. The laboratory space of the model is simplified to 2 zones on 2 floors (1 zone per floor) as we focus on the office spaces in this study. The office space has a total area of 3113.2 m<sup>2</sup> (i.e., 310.3 m<sup>2</sup>, 659.7 m<sup>2</sup>, 665.3 m<sup>2</sup>, 732.6 m<sup>2</sup>, and 745.3 m<sup>2</sup> of floor 1 to 5, respectively). The laboratory space

has a total area of 3165.6 m<sup>2</sup> (1570.3 m<sup>2</sup> of floor 1 and 1595.3 m<sup>2</sup> of floor 2). The building has a total of 1051.7 m<sup>2</sup> window area, of which 723.2 m<sup>2</sup> on the south façade, 103.5 m<sup>2</sup> on the east façade, and 225.0 m<sup>2</sup> on the north façade. Energy and thermal environment simulation is performed in this example case.



Figure 2-26. Pictures of Syracuse COE building (from Toshiko Mori Architect).

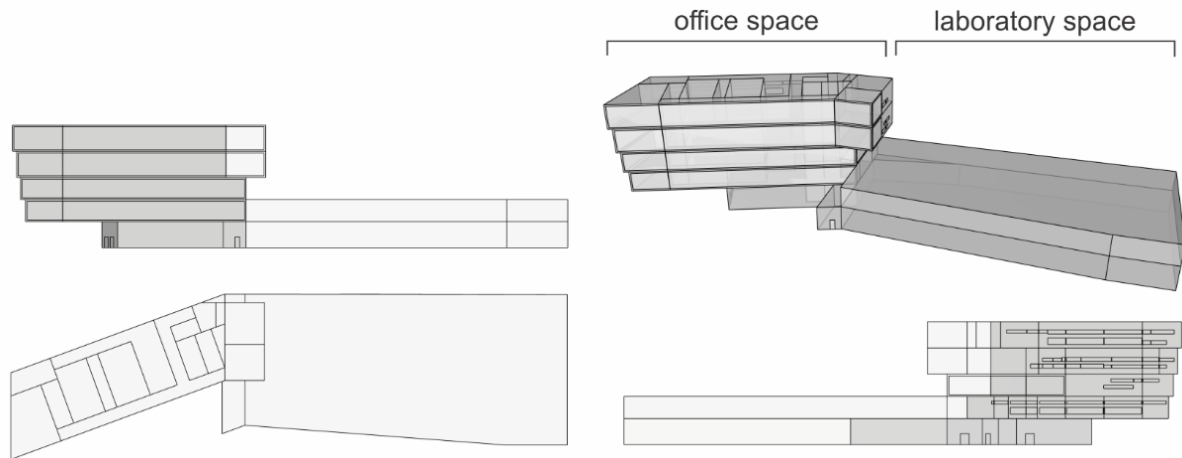


Figure 2-27. Schematic of the simulation 3D model of the building.

The same settings for Section 203 example of the building in Table 2-26 that is applied in the last section (Section 2.4.2) are applied to the whole building case here. The total energy use by the building during the whole simulation year is around 1,252,542 kWh. The average thermal comfort metrics across all zones of the building is around 82.2%. Monthly energy use for different types of applications (including heating, cooling, and fan energy of the HVAC system, lighting energy and equipment energy) is shown in Figure 2-28. Cooling energy consumption is higher than heating energy use. This is likely due to the high internal heat sources in the building, including occupant, lighting and equipment heat, as well as the higher heat gain from solar radiation through the large area of windows on the south façade. It means the design of glazing south façade can help to save considerable heating energy during the cold winter in Syracuse.

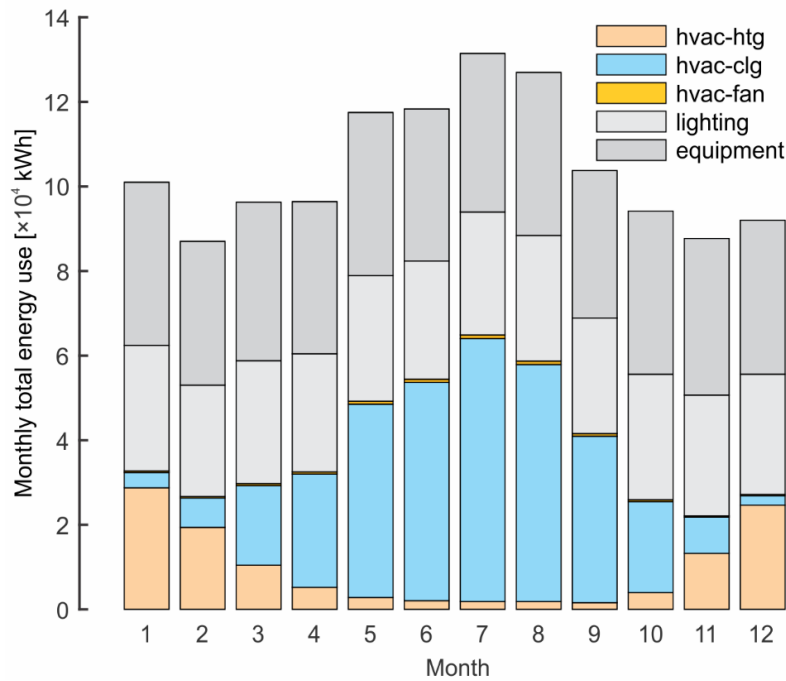


Figure 2-28. Monthly energy use for different types of applications by the building.

Room temperatures in different zones of the building are controlled by the ideal HVAC system. Figure 2-29 shows an example of the indoor room air temperature of the open office on the 3<sup>rd</sup> floor. The indoor temperatures are maintained well under the control of the ideal HVAC system. The distribution of room temperatures can be visualized by the GDS plugin in Rhino. Figure 2-30 demonstrates the room temperature distribution for the whole building on a typical summer day (June 21<sup>st</sup>) and a typical winter day (December 21<sup>st</sup>).

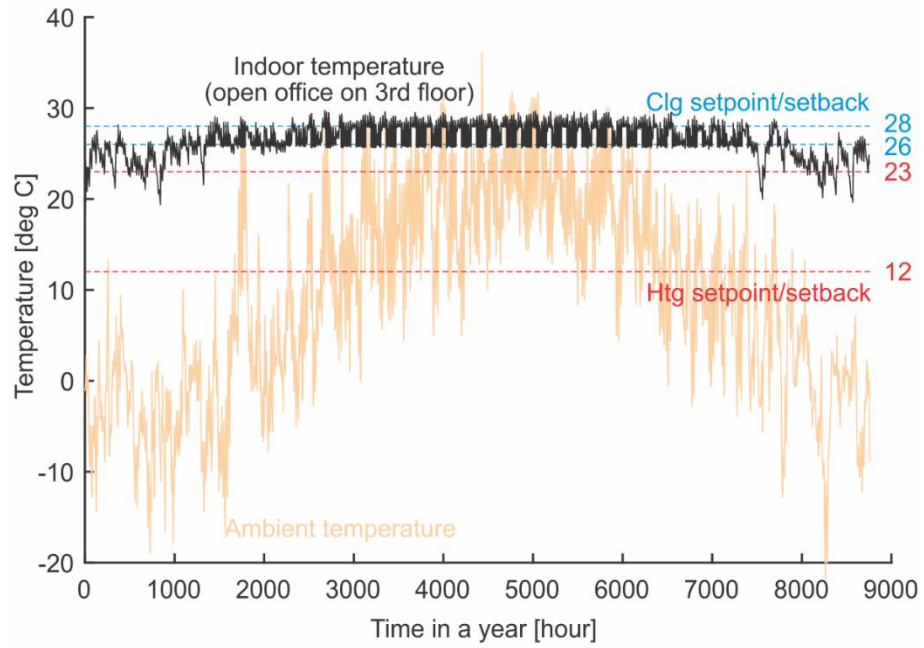
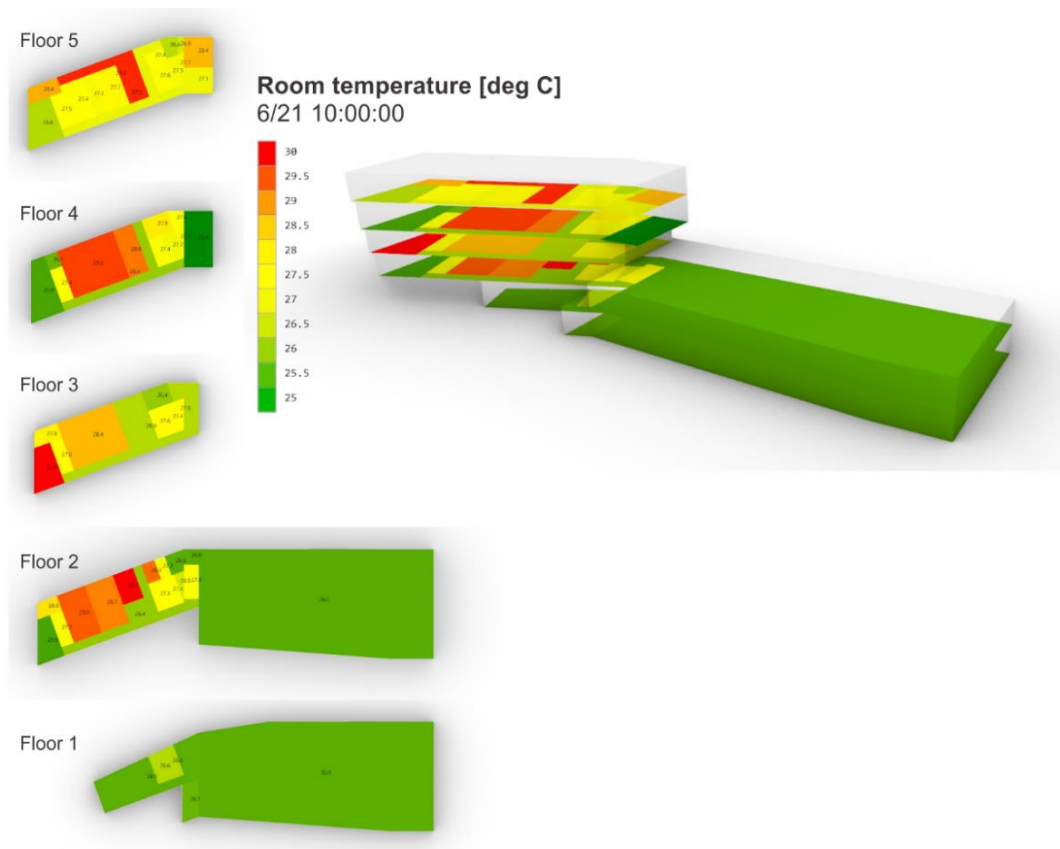
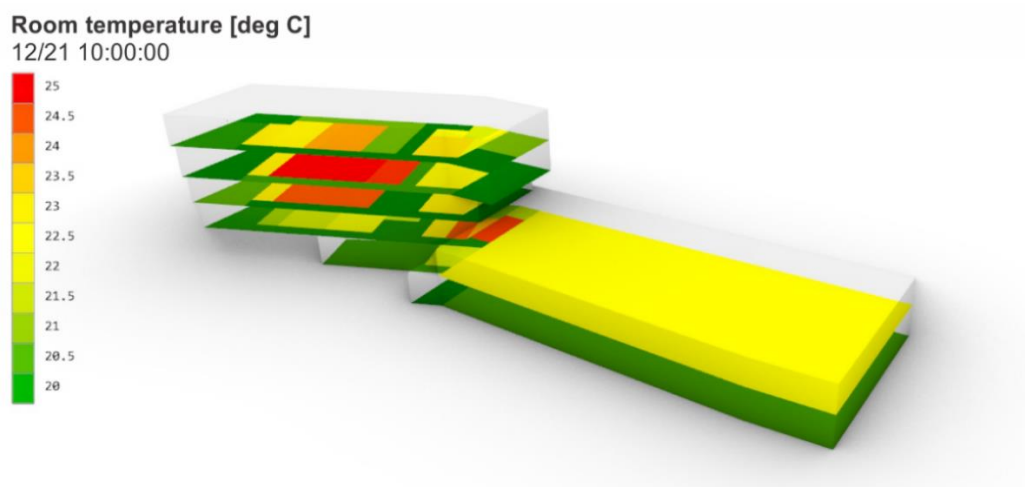


Figure 2-29. Whole-year indoor temperature of the open office on 3<sup>rd</sup> floor of the building.



(a) Room temperature distribution at 10AM on June 21<sup>st</sup>.



(b) Room temperature distribution at 10AM on December 21<sup>st</sup>.

Figure 2-30. Room temperature distribution of the whole building on Jun 21<sup>st</sup> and Dec 21<sup>st</sup>.

#### **2.4.4 Energy saving analysis for an integrated building retrofitting approach - a case study in cold/very cold climates**

Innovative building retrofitting strategies have been developed and implemented to achieve national goals of net-zero greenhouse gas emissions (White House 2021b, 2021a). Due to high heating demands in cold/very cold climates (McCabe et al. 2016), energy efficiency retrofits are desperately needed to reduce energy use. A whole-building retrofit solution with an envelope-HVAC integrated system has been developed to address this issue. The solution contains a novel highly-insulated exterior building envelope system that can be modularly attached to existing building enclosures, which includes a flashing solution for windows, doors, and penetrations for mechanical services, and an envelope-integrated HVAC solution that connects to an optimally-sized modular mechanical pod. Prototypes for the proposed retrofit solution have been developed and tested using facilities at Syracuse University including the Building Energy and Environmental Systems Laboratory (BEESL) and the Building Envelope Systems Testbed (BEST) (Krietemeyer et al. 2020). A diagram of the solution is shown in Figure 2-31.

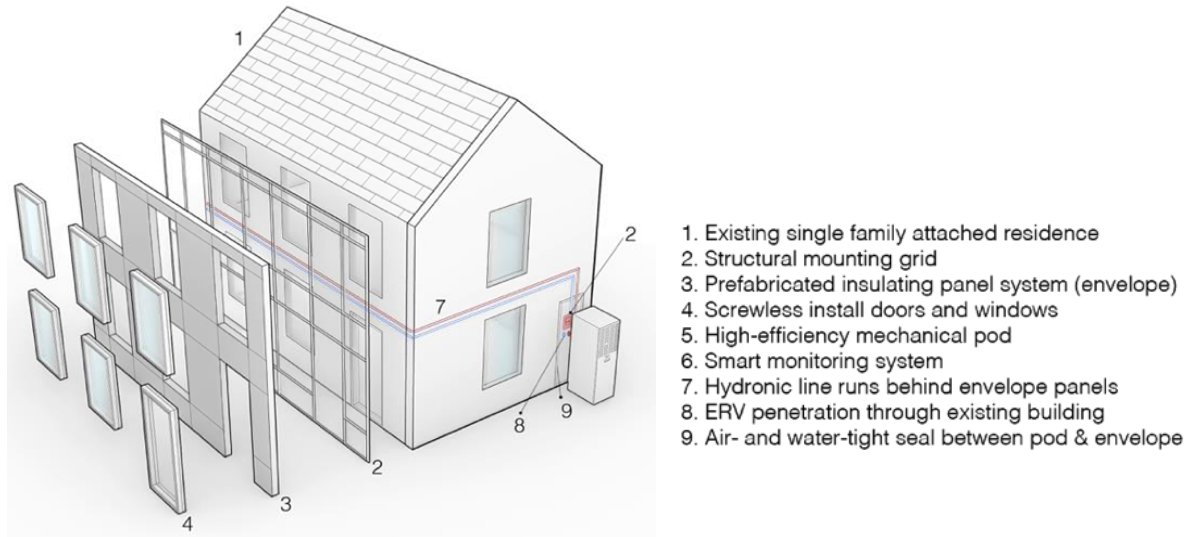


Figure 2-31. A diagram of the whole-building retrofit solution with integrated envelope-HVAC system (Krietemeyer et al. 2020).

Prefabricated retrofit envelope modules were developed in collaboration with Syracuse University, tkFabricate LLC, and Cocoon Construct Co. The envelope retrofit module consists of a 6-inch-thick lightweight sandwich panel that includes an expanded polystyrene (EPS) foam core, internal light-gauge galvanized steel members, and glass fiber reinforced cementitious lamina on all sides. The panel module assembly contains integrated structural studs and connection components that employ a "labyrinth" strategy to prevent air and water leakage. Additionally, an envelope-integrated HVAC system has been developed that enables intelligent monitoring and direct connections to a compatible modular mechanical pod for HVAC and DHW. The high-efficiency mechanical pod consists of an energy recovery ventilator (ERV) that uses smart occupancy-based control logic to ensure IAQ by maintaining CO<sub>2</sub> levels of less than 800ppm in buildings with airtightness of less than 1.0 L/s·m<sup>2</sup> exterior wall surface area at 50 Pa pressure differential. The mechanical pod offers various configurations that replace common heating, cooling, and DHW



appliances, such as fossil-fuel-driven forced air and hydronic systems, electric resistance heating, and window air conditioners, with a single high-efficiency air-to-water cold-climate heat pump and ERV. This heat pump and ERV have a core recovery efficiency of 88%. Therefore, the proposed retrofit solution can provide higher thermal insulation, better airtightness, and higher heating/cooling/DHW efficiency.

The energy performance of the retrofit solution was originally tested in EnergyPlus. In this study, the GDS approach is used to evaluate its energy performance. The solution is applied to the BEST house at Syracuse University, which is a two-floor house with replaceable envelope panels (Figure 2-32). In the field experiment, a small test room in the southeast corner of the first floor was built up with the retrofit solution implemented. Prefabricated envelope panels were mounted on the exterior walls of the test room. The mechanical pod was integrated to provide air conditioning and DHW supply to the test room. In this study, the retrofit solution is applied to the whole building to simulate its performance on a realistic-scale building. The retrofit envelope panels were mounted outside the existing exterior walls with a window on the east side of the test room. The energy-efficiency mechanical pod was used to provide heating and cooling demand. DHW is not simulated in this study. A reference building is created according to the median thermal EUI of the single-family attached building type from Building America House Simulation Protocols developed by NREL (Wilson et al. 2014). Some key settings of the reference case and the proposed case related to the retrofit strategy are shown in Table 2-28. Other settings are adapted from the study cases in the above sections. Energy use due to DHW is not simulated in the present study. The air conditioning efficiency of the retrofit module (mechanical pod) is assumed to be 3.75 times more efficient than the conventional unit based on the efficiency in Table 2-28.

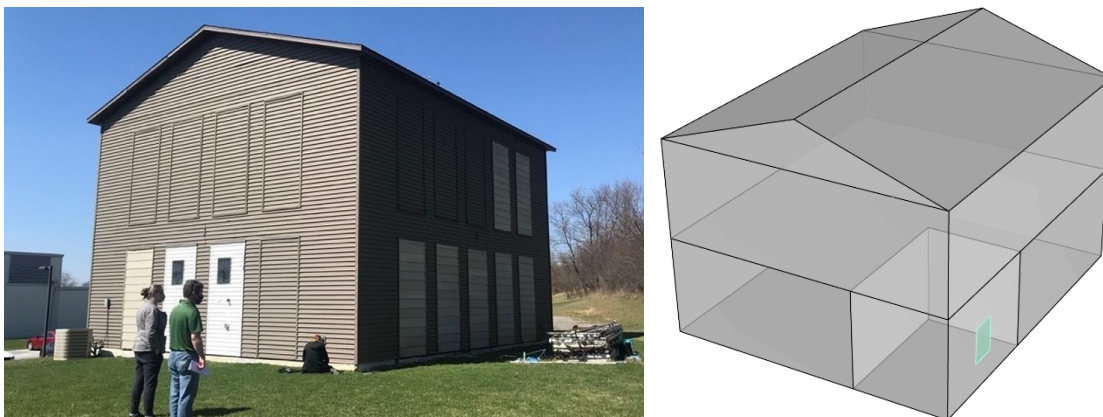


Figure 2-32. A picture and brief illustration of the BEST house.

Table 2-28. Summary of simulated conditions of the reference case and the proposed case.

Retrofit strategies	Pre-retrofit reference	Post-retrofit case
Airtightness	2.2 ACH <sub>50</sub>	1.05 ACH <sub>50</sub> <sup>b</sup>
Insulation - Exterior Wall	R-17 <sup>a</sup>	R-30
Insulation - Roof	R-9 <sup>a</sup>	-
Heating Equipment Efficiency	80 AFUE <sup>a</sup>	3 COP <sup>c</sup>
Cooling Equipment Efficiency	9.1 EER <sup>a</sup>	23 EER <sup>c</sup>
DHW Efficiency	0.56 EF <sup>a</sup>	2.43 COP <sup>c</sup> (not simulated)

<sup>a</sup> 2014 Building America House Simulation Protocols by NREL (Wilson et al. 2014).

<sup>b</sup> Based on the target airtightness level for the retrofitting system.

<sup>c</sup> Estimated per the Energy POD design specifications (Krietemeyer et al. 2020).

The simulation was performed on the same computer as presented in Section 2.4.1. The simulation finished in 18.4 seconds. The simulation results show that the unit energy use after retrofitting is around 173.7 kWh/m<sup>2</sup> per floor area compared to 314.4 kWh/m<sup>2</sup> before retrofitting. A 44.8% improvement in energy efficiency is achieved. The earlier work by EnergyPlus suggested an improvement of 52.2% in energy efficiency for heating and cooling, which is comparable to the result here. Therefore, the proposed retrofit solution has a significant potential of energy efficiency improvement.

## **2.5 Conclusions**

This Chapter presented the major part of this work. A review of main BPS models was performed. Considering the requirement of modeling response and reliability for early-stage green building design, reduced-order physics-based models were applied in the GDS tool. Albeit not applied in the present work, reduced-order models have potential to integrate with measurement data to build hybrid models for improving simulation accuracy for buildings in actual scenarios. RC-structured physical models that represent building thermal, air and mass balance equations were created. Building modules were formatted to represent building systems and strategies across different scales. Building performance, including energy efficiency, thermal comfort, DALYs due to indoor contaminant exposure, and infection risks due to airborne transmission of respiratory diseases, were calculated. The simulation models were implemented in Matlab first and developed as a usable tool in Rhino Grasshopper. A plugin was developed in C# to integrate proposed BPS models

and geometry modelling functions in Rhino/Grasshopper. Multiple components were created in Rhino Grasshopper to comply with proposed building modules. The developed plugin is the main platform of the GDS approach.

Three study cases were performed using the developed platform, including a DOE prototype of small office building, a section of a realistic green building, Syracuse COE headquarter building, and a retrofitting case of an innovative integrated envelope-HVAC solution. A variety of energy-efficiency and IAQ improvement strategies were applied in the small office prototype building to estimate their performance. Building strategies usually have very variable impacts on energy and IAQ performance. Some strategies may improve certain aspects of building performance while impairing others. Therefore, it is important to find out the relatively better or optimal combination of the application of green building strategies. A simple application of building performance optimization was conducted and found that under the simulated conditions, the optimal design of the proposed small office prototype can save up to 27.8% energy use while mitigate more than 99% infection risk compared to the reference case. Therefore, it reveals that the optimization of green building design using the proposed approach has high potential of energy and IAQ improvement.

A similar analysis was performed to the Syracuse COE case. It analyzed the performance of some HPB strategies in an existing green building. It suggested that it is not recommended to implement all proposed technologies in the target building as that would impair thermal comfort in some scenarios. In order to achieve a better performance and save the building cost for the study case, higher airtightness, elevated insulation, and controlled lighting strategy are recommended to be

used in the design. It can save around 14% of total energy use and elevate over 3% of people satisfied with the thermal comfort in corridor but have a slight decrease in the thermal comfort in room.

The retrofitting case analyzed the performance of an innovative retrofitting strategy in cold/very cold climates. The solution contains a prefabricated envelope with higher thermal insulation and better airtightness. An energy-efficiency mechanical pod is integrated with the envelope system. It shows that a 44.8% improvement in energy efficiency can be achieved using the proposed solution compared to a reference case.

### **3. Database of green building technologies**

#### **3.1 Review of state-of-the-art research**

##### **3.1.1 Whole-building energy and IAQ database**

Many public databases in the building sector have been established. Table 3-1 shows some major building performance databases associated with the research in this work. Some of these databases contain building parameters and performance metrics like energy use, while others only have general information for whole buildings or building components such as envelope materials without performance metrics data. The datasets from these databases are usually used to extract the information of different buildings or building components (e.g., understanding the geometry information of a certain type of building), and train data-driven models for building performance estimation and optimization. The U.S. DOE funded Building Performance Database (LBNL 2011) is likely the most widely used database for building energy analysis, which contains 1,115,196

datasets from residential and commercial buildings. The datasets contain information including climate and location, building parameters (e.g., built year, operating hours, floor area), building systems (e.g. lighting, cooling/heating system, glazing, wall insulation, air flow control), occupant features (e.g. number, density), green certifications (e.g. LEED score, Energy Star Rating), energy use intensity, and carbon emission. It also has a web-based tool that can be used for statistical analyses and demonstrations. The datasets were obtained from various sources, including public sectors (e.g. California Energy Commission, New York City, EPA Energy Star, EIA CBECS and RECS databases, and DOE Better Buildings Challenge) and private sectors (e.g. USAA Real Estate Company, and Dayton Residential) (Mathew et al. 2015).

U.S. Energy Information Administration (EIA) has performed periodic surveys on energy consumption for residential and commercial buildings. The obtained datasets are included in the DOE Building Performance Database. The Residential Energy Consumption Survey (RECS) (U.S. EIA 2020a) provides detailed information about energy usage in U.S. homes. RECS is a multiyear effort consisting of a household survey, data collection from household energy suppliers, and end-use consumption and expenditures estimation. The housing characteristics in the latest survey were collected in late 2020 and early 2021 (released in July 2022) from nearly 18,500 households to represent the 123.5 million housing units that are occupied as a primary residence (U.S. EIA 2020a). The datasets include building characteristics such as structural and geographic characteristics, square footage, appliances, electronics, lighting, space heating, air conditioning, water heating, household demographics, and energy use. The Commercial Building Energy Consumption Survey (CBECS) (U.S. EIA 2020b) focuses on commercial buildings in U.S. The latest CBECS contains 6,436 records of commercial buildings that represent an estimated 5.9

million total commercial buildings in the U.S. (as of 2018). Building characteristics data tables of CBECS include number of workers, ownership and occupancy, structural characteristics, energy sources and uses, energy related building features, and more (U.S. EIA 2020b). For both surveys, full datasets and comprehensive reports that contain statistical tables representing the distribution of collected data are released. Similar building energy surveys have been performed by many agencies across countries. For example, the National Energy Use Database (NEUD) (NRCan 2021) is established by the Natural Resources Canada (NRCan) through several nationwide surveys including the Survey of Household Energy Use (SHEU), the Survey of Commercial and Institutional Energy Use (SCIEU), the Survey of Energy Consumption of Multi Unit Residential Buildings (SECMURBs), the Survey of Energy Consumption of Arenas (SECA), and the Industrial Consumption of Energy (ICE) Survey. The database contains building characteristics and energy use.

The Building Energy Benchmarking Data was obtained through the Building Energy Benchmarking Program hosted by local governments, which requires owners of non-residential and multifamily buildings to track energy performance and annually report to the local agency. The program has been performed in more than 100 cities in the U.S. (ACEEE 2022). Data of some cities has been posted and can be accessed online, e.g. 3,628 datasets in Seattle, 3,939 datasets in New York City, and 17,389 datasets in California (as of 2020) (California Energy Commission 2022; Department of Finance of New York City 2022; Office of Sustainability & Environment of Seattle 2022). The datasets usually contain location, building characteristics (e.g. primary use, floor number, and built year), neighborhood information, energy use, carbon emission, and green certification information.

The Building Data Genome Project 2 Database (C. Miller et al. 2020) an open database of 3,053 energy meters from 1,636 non-residential buildings with a range of two full years (2016 and 2017) at an hourly frequency (17,544 measurements per meter resulting in approximately 53.6 million measurements). These meters were collected from 19 sites across North America and Europe, with one or more meters per building measuring whole building electrical, heating and cooling water, steam, and solar energy as well as water and irrigation meters. Part of this data was used in the Great Energy Predictor III (GEPIII) competition hosted by ASHRAE on Kaggle. The datasets include information like climate and location, weather data, building characteristics (e.g., floor area, built year, floor number, etc.), occupant number, energy use, and green certification of buildings.

The energy performance certificates (EPCs) are a rating scheme to summarize the energy efficiency of buildings (Energy Saving Trust 2022). The Energy Performance of Buildings Data (DLUHC 2022) provides datasets of EPCs in the U.K. issued up to March 2022, including data from 22,710,206 domestic and 1,059,491 non-domestic EPCs. The datasets include climate and location, building characteristics (e.g., floor area), energy use, carbon emission, and green certification. The Low Energy Building (LEB) Database (LEB 2010) is a repository of green building information created by AECB (U.K.) to help inform the planning and development of new build and refurbishment of green buildings. The repository contains 500 green building projects in the U.K. with detailed project descriptions (e.g. location, occupation date, construction type, and floor area), building service systems (e.g. lighting, heating/cooling, ventilation, and other appliances), building construction characteristics (e.g. roof, wall, floor, glazing descriptions and U-values), green design strategies (e.g. heating/cooling, ventilation, renewable energy, insulation,



daylighting, and airtightness strategies), energy use and renewable energy generation by measurement or modelling, and green certification. Some green building certification/rating systems such as LEED, also have repositories that contain green building projects with basic building information and green certification/rating levels. For example, the LEED database contains over 86,000 certified residential and commercial projects across the world, each of which provides basic project descriptions (e.g., project type, location, and floor area) and the score that project earned in different evaluation aspects including sustainable site, water efficiency, energy and atmosphere, material and resources, indoor environmental quality, innovation, and regional priority credits. However, some key performance metrics like energy use or thermal comfort are not provided, therefore, are less commonly used by quantitative analyses. But some individual studies have performed surveys, field measurements, or modellings to obtain quantitative metrics regarding actual building performance for some LEED-certified projects (Newsham et al. 2009; Scofield and Doane 2018; Turner et al. 2008; P. Wu et al. 2016), which can be adopted to establish a LEED-certified project database with more quantitative features.

The ASHRAE Global Thermal Comfort Database II (Földváry Ličina et al. 2018; Parkinson et al. 2022) is comprised of field studies of thermal comfort from around the world, containing a total of 109,033 entries. The datasets are collected through instrumental measurements and subjective surveys, consisting of information of climate and location, season and date, building characteristics (e.g. building type, window, door), system configurations (e.g. fan, heating/cooling systems), occupant information (e.g. gender, activity level, clothing), indoor variables (e.g. temperature, humidity), and thermal comfort metrics (e.g. PMV, PPD, thermal and air movement acceptability, and thermal and air movement preference). The ASHRAE Global Occupant Behavior Database

(B. Dong et al. 2022) is a comprehensive global database about building occupant behavior. The database contains 34 field-measured building occupant behavior sources collected from 15 countries and 39 institutions across 10 climatic zones covering various building types in both commercial and residential sectors. The database includes occupancy patterns (i.e., presence and people count), occupant behaviors (i.e., interactions with devices, equipment, and systems such as door, window, fan, lighting, shading systems), indoor variables (e.g., temperature, humidity, and thermal comfort metrics such as PMV), and measurement technology (i.e. sensor information).

The Global Household Air Pollution (HAP) Database (Shupler et al. 2018), commissioned by the WHO and sponsored by the Natural Sciences and Engineering Research Council (NSERC) of Canada, provides an organized summary of data reported in the literature describing HAP microenvironments, methods and measurements associated with kitchen activities. The database contains 1290 datasets of field measurements from 196 studies published across 43 countries through 2016. The database includes general information of the studied household (e.g. location, building design, ventilation, kitchen location and type, primary cooking fuel, heating fuel, and stove type), occupant characteristics (e.g. occupant number, age, gender, and activities), field measurement data of PM<sub>2.5</sub> and CO, simulated levels of PM<sub>2.5</sub> and CO, health assessments evaluated by measurement data, and other health metrics derived from measurement or modelling. Many other studies, particularly literature reviews, have collected similar IAQ datasets of field measurements or modellings, including quantitative metrics of indoor pollutant concentrations such as particles, CO<sub>2</sub>, ozone and VOCs, and health effect metrics (K. K. Lee et al. 2020; M. Lee et al. 2021; Lim et al. 2022; Puttaswamy et al. 2021; Shupler et al. 2020, 2022; Z. Yang et al. 2018). However, most of these studies do not provide datasets as detailed/comprehensive as the

WHO Global HAP Database. Some key variables like building characteristics are not included as their studies focus on pollutant exposure or health effect assessments. Therefore, those studies are not considered by the present work.

Numerous datasets of COVID-19 outbreaks have been reported around the world. However, most of these datasets did not take building characteristics into consideration. Colorado Department of Public Health and Environment (CDPHE) has released COVID-19 outbreak data reported in public agencies with information of location, outbreak date, agency name (building name), building function (e.g. office, school, or healthcare), infection and death number (CDPHE 2020; Shen, Kong, Dong, et al. 2021b). The database includes data of 9,093 resolved outbreaks in Colorado. Although detailed building characteristics such as floor area and building construction are still missing, some of this information can be obtained based on online searching since the agency name of each outbreak is reported and thus can be found online. A database with more comprehensive information is therefore potential to be established. But the current database is still useful for data analyses (Shen, Kong, Dong, et al. 2021b).

### **3.1.2 Building component/system database**

Some databases or libraries of building components and systems have been established (Table 3-2). These databases or libraries are usually developed to be used by BPS programs (energy and IAQ). Performance metrics are usually not included as a part of the building component or system. The Building Component Library (NREL 2021) is a collection of building data used to create building energy models without performance metrics included. The data are broken down into

components that represent parts of a building and measures that describe changes made to a building. The datasets are structured following the format of EnergyPlus input data file (IDF) or OpenStudio model file (OSM). The information is from real practices, which are contributed by multiple individuals or agencies. The IEA Annex 68 compiled a VOC emission database of some common building products with measured data (Qin et al. 2020), including information such as material diffusion coefficient, partition coefficient, and initial concentration. Similar databases with VOC emission parameters include the PANDORA database (M. Abadie and Blondeau 2011) and building material emission library in CHAMPS-BES (Grunewald and Nicolai 2005). The reaction probabilities of common building materials with ozone have also been compiled as a database that can be used for indoor ozone chemistry modelling (Shen et al. 2017; Shen and Gao 2018).

Table 3-1. Representative building performance databases associated with this study (as of 2022).

Database	Sponsor	Description	Key variables
Building Performance Database (BPD) (LBNL 2011)	DOE	<p>Dataset size: 1,115,196 datasets</p> <p>Building type: Residential and commercial</p> <p>Scale: Whole building</p> <p>Performance metrics: green certifications, energy use intensity, carbon emission (annual total)</p> <p>Data acquisition: survey</p>	<p>Climate and location, building parameters (e.g. built year, operating hours, floor area), building systems (e.g. lighting, cooling/heating system, glazing, wall insulation, air flow control), occupant features (e.g. number, density), green certifications (e.g. LEED score, Energy Star Rating), energy use intensity, and carbon emission</p>
Residential Energy Consumption Survey	EIA	<p>Dataset size: 18,496 datasets</p> <p>Building type: Household</p> <p>Scale: Whole building</p>	<p>Structural and geographic characteristics, square footage, appliances, electronics, lighting,</p>

(RECS) datasets (U.S. EIA 2020a)		Performance metrics: energy use (annual total)  Data acquisition: survey	space heating, air conditioning, water heating, household demographics, and energy use
Commercial Building Energy Consumption Survey (CBECS) datasets (U.S. EIA 2020b)	EIA	Dataset size: 6,436 datasets  Building type: Commercial  Scale: Whole building  Performance metrics: energy use (annual total)  Data acquisition: survey	Number of workers, ownership and occupancy, structural characteristics, energy sources and uses, energy related building features, etc.
Building Energy Benchmarking Program Data (California Energy Commission 2022; Department of Finance	Local governments	Dataset size: City/state-dependent (e.g. 3,628 datasets for Seattle, 3,939 datasets for New York City, and 17,389 datasets for California, as of 2020)	Location, building characteristics (e.g. primary use, floor number, and built year), neighborhood information, energy use, carbon

of New York City		Building type: Large commercial and multifamily buildings	emission, and green certification information
2022; Office of Sustainability & Environment of Seattle		Scale: Whole building	
2022)		Performance metrics: Energy use, green certification (annual)	
		Data acquisition: survey	
Building Data Genome Project 2 Database (C. Miller et al. 2020)	ASHRAE	Dataset size: 53.6 million measurements from 1,636 commercial buildings	Climate and location, weather data, building characteristics (e.g. floor area, built year, floor number, etc.), occupant number, energy use, and green certification
		Building type: Commercial	
		Scale: Whole building	
		Performance metrics: Energy use (time-sequence), green certification	
		Data acquisition: survey, field measurement	

Energy Performance of Buildings Data (DLUHC 2022)	DLUHC (U.K.)	Dataset size: Data from 22,710,206 domestic and 1,059,491 non-domestic EPCs  Building type: Residential and commercial  Scale: Whole building  Performance metrics: Energy use, carbon emission, green certification (annual)  Data acquisition: survey	Climate and location, building characteristics (e.g. floor area), energy use, carbon emission, and green certification
Low Energy Building (LEB) Database (LEB 2010)	AECEB (U.K.)	Dataset size: 500 building projects  Building type: Residential and commercial  Scale: Whole building  Performance metrics: Energy use, renewable energy generation, carbon emission, green certification (annual)  Data acquisition: survey, modelling	Detailed project descriptions (e.g. location, occupation date, construction type, and floor area), building service systems (e.g. lighting, heating/cooling, ventilation, and other appliances), building construction characteristics (e.g. roof, wall, floor, glazing descriptions)



			and U-values), green design strategies (e.g. heating/cooling, ventilation, renewable energy, insulation, daylighting, and airtightness strategies), energy use, renewable energy generation, and green certification
LEED certificated building datasets (USGBC 2022)	USGBC	Dataset size: 86,193 datasets  Building type: Residential and commercial  Scale: Whole building  Performance metrics: Green certification (LEED scores)  Data acquisition: survey	Basic project descriptions (e.g. project type, location, and floor area), and LEED rating scores for various evaluation aspects, including sustainable site, water efficiency, energy and atmosphere, material and resources, indoor environmental quality, innovation, and regional priority credits

ASHRAE Global Thermal Comfort Database II (Földváry Ličina et al. 2018; Parkinson et al. 2022)	ASHRAE	Dataset size: 109,033 datasets  Building type: Residential and commercial  Scale: Whole building  Performance metrics: Thermal comfort (daily)  Data acquisition: field measurement (real scenarios), subjective survey (questionnaire)	Climate and location, season and date, building characteristics (e.g. building type, window, door), system configurations (e.g. fan, heating/cooling systems), occupant information (e.g. gender, activity level, clothing), indoor variables (e.g. temperature, humidity), thermal comfort metrics based on instrumental measurements or subjective surveys (e.g. PMV, PPD, thermal and air movement acceptability, and thermal and air movement preference)
ASHRAE Global Occupant Behavior	ASHRAE	Dataset size: Datasets from 34 sources  Building type: Residential and commercial	Occupancy patterns (i.e., presence and people count), occupant

Database (B. Dong et al. 2022)		<p>Scale: Whole building</p> <p>Performance metrics: Thermal comfort (time-sequence)</p> <p>Data acquisition: field measurement</p>	<p>behaviors (i.e., interactions with devices, equipment, and systems such as door, window, fan, lighting, shading systems), indoor variables (e.g. temperature, humidity, and thermal comfort metrics such as PMV and thermal sensation vote), and measurement technology (i.e. sensor information)</p>
Global household air pollution database (Shupler et al. 2018)	WHO, NSERC	<p>Dataset size: 1,290 datasets from 196 sources</p> <p>Building type: Residential</p> <p>Scale: Whole building</p>	<p>General information of the studied household (e.g. location, building design, ventilation, kitchen location and type, primary cooking fuel, heating fuel, and stove type), occupant characteristics (e.g. occupant number, age, gender, and</p>

		Performance metrics: PM2.5 and CO concentration and health assessments (average over the test period)	activities), field measurement data of PM2.5 and CO, simulated levels of PM2.5 and CO, health assessments evaluated by measurement data, and other health metrics derived from measurement or modelling
COVID-19 indoor outbreak database (CDPHE 2020; Shen, Kong, Dong, et al. 2021b)	CDPHE	Dataset size: 9,093 datasets (resolved cases)  Building type: Residential and commercial  Scale: Whole building  Performance metrics: Infection and death number  Data acquisition: Survey	Location, outbreak date, agency name (building name), building function (e.g. office, school, or healthcare), infection and death number

Table 3-2. Building component and system database.

Database	Sponsor	Description	Key variables
Building Component Library (BCL) (NREL 2021)	DOE	<p>Dataset size: 34,096 datasets</p> <p>Building type: Residential and commercial</p> <p>Scale: Building component or system</p> <p>Performance metrics: NA</p> <p>Data acquisition: Survey, practice</p>	<p>Detailed characteristics of building components and systems in IDF (for EnergyPlus) or OSM (for OpenStudio) formats</p>
Material VOC emission database (Qin et al. 2020)	IEA	<p>Dataset size: Data of 5 common materials</p> <p>Building type: Residential and commercial</p> <p>Scale: Building material</p> <p>Performance metrics: NA</p> <p>Data acquisition: Laboratory measurement</p>	<p>VOC emission parameters of materials (including in-material diffusion coefficient, partition coefficient, and initial concentration)</p>

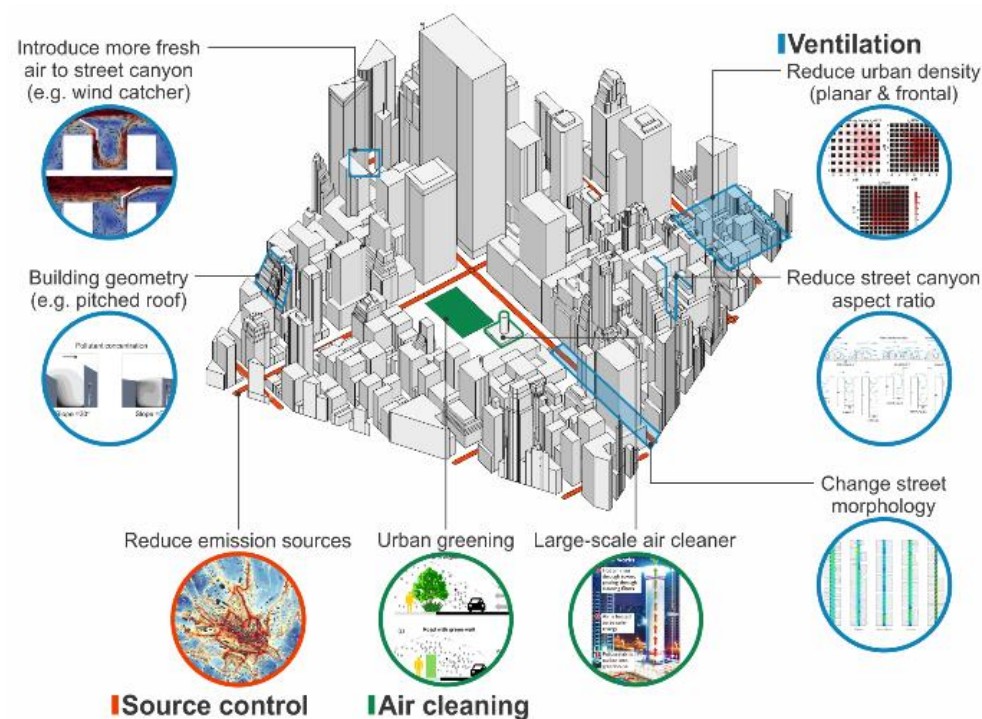
Material ozone / deposition rate database (Shen et al. 2017; Shen and Gao 2018)	Dataset size: Data of 54 common materials  Building type: Residential and commercial  Scale: Building material  Performance metrics: NA  Data acquisition: Laboratory measurement	Reaction probability of the material with ozone
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## **3.2 Database of green building technologies**

### **3.2.1 Green building technology definition and collection**

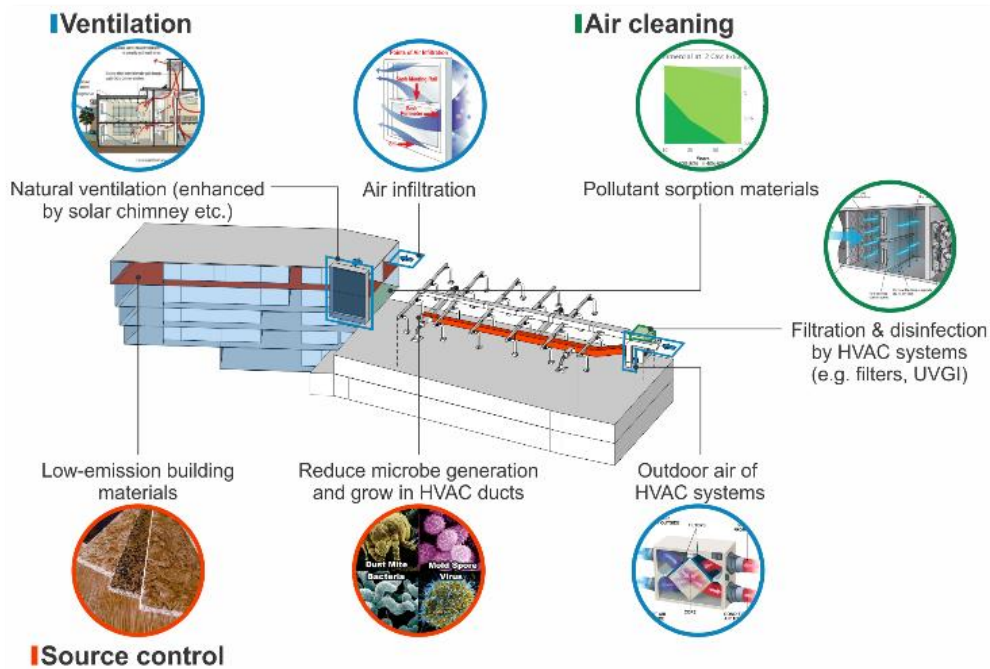
There have been numerous green building technologies developed to address building energy and IAQ issues, so that the technologies can be divided into two categories regarding their major purposes, i.e. energy-efficiency technologies and IAQ-enhance technologies (Cao et al. 2016; Shen, Kong, Dong, et al. 2021a, 2021b). Some technologies are designed to save energy use and improve IAQ simultaneously, thus can be included in both categories. Green building technologies can also be categorized based on their energy use condition. Passive strategies are the building technologies that do not require or require minimal active energy inputs, which are usually more energy-efficient for application in buildings. Active technologies are the technologies that require energy consumption. Those passive technologies actually have a much longer history associated with their use in buildings than active strategies, before electricity was utilized by human beings (X. Li et al. 2017). Early passive technologies, such as cave dwellings (utilizing thermal insulation and thermal mass of the cave), kang (a domestic heating system used in rural northern China) (S. Yan et al. 2020; Zhuang et al. 2009), wind catcher (a system introduces cooled outdoor air to indoor space in Middle East) (Masrour et al. 2012; Montazeri et al. 2010), etc. have existed for hundreds or thousands of years for improving indoor thermal or air quality environments, and some of them are still being utilized in many modern buildings. Although passive technologies usually consume minimal energy, their efficiencies are sometimes limited and insufficient for modern buildings. Active systems, conversely, consume a certain quantity of energy use, but can condition the indoor air more efficiently.

For IAQ improvement technologies, typical IAQ improvement technologies include three categories, i.e., source control, ventilation, and air cleaning. IAQ technologies at different scales can have very different characteristics and efficiencies. Figure 3-1 shows the typical IAQ strategies at different scales of building/urban environment, from urban scale, building scale, room scale to personal microenvironment (J. J. Zhang et al. 2022). During COVID-19 pandemic, many of these IAQ strategies have been applied to mitigate the airborne transmission of SARS-CoV-2. Some emerging technologies were also proposed or utilized, such as the needlepoint bipolar ionization technology. An earlier study evaluated the effectiveness of different IAQ strategies in mitigating airborne transmission of COVID-19 (Shen, Kong, Dong, et al. 2021a).

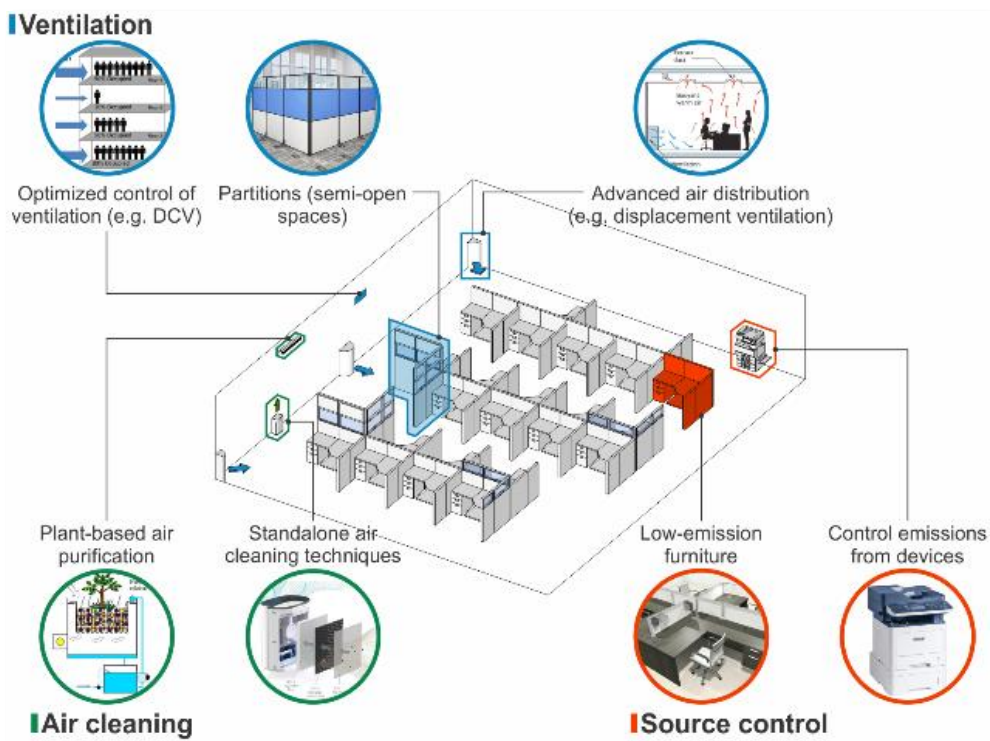


(a) Urban-scale strategies.

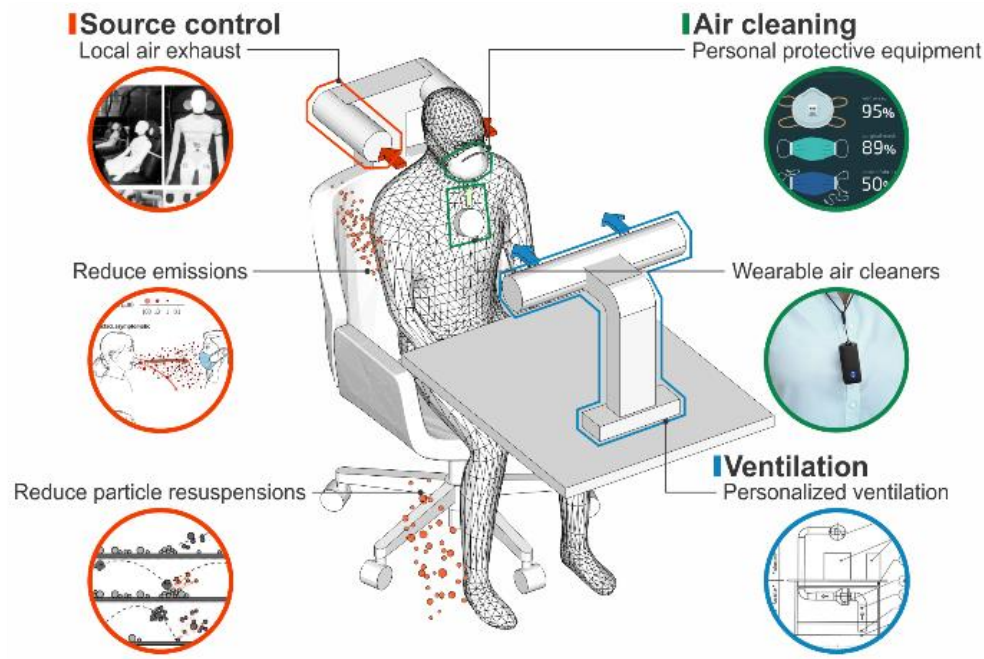




(b) Building-scale strategies.



(c) Room-scale strategies.



(d) Microenvironment-scale strategies.

Figure 3-1. IAQ control strategies at multiple scales (adapted from (J. J. Zhang et al. 2022)).

### 3.2.2 Baseline case definition

In order to evaluate the effectiveness of various energy efficiency and IAQ strategies, it is necessary to define a baseline case as a reference. A local reference building is defined as a building, with its construction complying with the local standards/codes or adopting the local best practice. The local best practice of building design can be accessed through the local building documents, reports, cases, or available datasets, e.g. the Building Performance Database (LBNL 2011). When the local best practice is unavailable or does not meet the local building standards and codes, then the local standards and codes will be adopted to define the reference building. Many standards or codes have developed reference criteria for various types of buildings. For

example, DOE (U.S. DOE 2020) has developed commercial reference buildings of 16 building types that represent approximately 70% of the commercial buildings in the U.S., according to a report published by the National Renewable Energy Laboratory (NREL) (Deru et al. 2011). The reference buildings provide complete descriptions for whole building energy analysis using EnergyPlus. The Pacific Northwest National Laboratory (PNNL) modified the DOE commercial prototype building models (U.S. DOE 2013) to accommodate the ASHRAE Standard 90.1 (ASHRAE 2019c) and International Energy Conservation Code (IECC) (ICC 2009). The PNNL commercial prototype buildings (U.S. DOE 2013) cover all DOE reference building types (with the exception of supermarkets), and also an additional prototype representing high-rise apartment buildings, resulting in 16 commercial building types in 19 climate locations. For residential buildings, PNNL uses two base prototypes to simulate single-family detached houses and multi-family low-rise apartment buildings (U.S. DOE 2013). These prototypes are then modified to accommodate four different heating system types and four foundation types typically found in new residential construction. For single-family houses in the U.S., NREL has developed the benchmark building for each climate zone (i.e. Building America B10 Benchmark (Wilson et al. 2014)), which is consistent with the 2009 IECC (ICC 2009). Besides, ASHRAE standards (ASHRAE 2018, 2019b) defined the design criteria like envelope construction and ventilation requirement for single-family house in each climate zone, which should also be satisfied by the local reference house. The IEA Annex 68 project developed a detailed procedure along with an example for defining a local reference building for both IAQ and energy performance evaluation (Zhenlei Liu et al. 2017; Qin and Zhang 2020). Baseline settings of public buildings in terms of airborne infection risk estimation and IAQ evaluation were also presented recently (Shen, Kong, Dong, et

al. 2021a). In this study, the DOE reference small office case was used as the reference case, which has been introduced in earlier sections.

For retrofitting projects, the existing conditions before the retrofitting can be selected as the baseline for comparison as in (Krietemeyer et al. 2020). This case was also discussed as an application example of the GDS in Chapter 2.

### **3.2.3 Database**

A review was performed to collect these energy-efficiency and IAQ-improvement building technologies (Cao et al. 2016; Shen, Kong, Dong, et al. 2021a, 2021b; J. J. Zhang et al. 2022). Table 3-3 shows the major green building strategies collected from literature review. Table 3-4 provides more detailed information of airborne infection mitigation strategies.

Table 3-3. Typical strategies to improve building energy and IAQ improvements.

Category			Strategy
Energy efficiency	Passive technologies	Building envelope	Higher thermal insulation (e.g. vacuum insulation panel (VIP), prefabricated insulation panel)
			Ventilated/double-skin walls/roofs/floors to increase insulation or to enhance convective heat loss
			Passive solar thermal walls (e.g. Trombe wall)
			Daylight harvesting
			Energy-saving glazing technologies (e.g. double glazed window with argon gas, low-E coating)
			Passive shading technologies
			Low infiltration (airtightness enhancement)
			Solar-reflective/cool roofs

		Green roofs
		Evaporative roof cooling
Passive heating, cooling and ventilation		Nighttime ventilation
		Earth-to-air heat exchange
		Natural ventilation
		Solar chimney (vented Trombe wall)
Thermal storage		Phase-change materials (PCM)
		Thermal mass (e.g. concrete)
Service systems	HVAC	Evaporative cooling
		Active thermal storage
		Heat recovery system
		Radiant heating/cooling

	VAV/VRF
	Demand-controlled ventilation (DCV)
	Optimized control of ventilation
DHW	Solar water heater
	Solar-assisted heat pump system
	Combined cooling, heating and power
Lighting	LEDs
	Energy efficient lighting
	Active shading technologies
	Demand response lighting
Appliances	Energy-efficient appliances
	Demand response appliances

		Smart control strategies	Smart control algorithms to operate active systems in a more efficient way (e.g. MPC)
	Renewable energy generation	Solar energy	PV/building integrated PV
			Hybrid PV-thermal
		Wind energy	Wind turbine
		Geothermal energy	Ground source heat pump
		Bioenergy	Biomass heater
IAQ improvement	Source control	Outdoor pollutants	Reducing outdoor pollutants (traffic emissions)
			Greening
		Indoor pollutants	Low-emission building materials/furniture
			Local air exhaust
		Occupant pollutants	occupancy control/screening
			face covering



Ventilation	Passive ventilation	Natural ventilation
		Solar chimney
		Air infiltration
	HVAC system	Increasing outdoor air supply
		High-efficiency filters (e.g. HEPA)
	Optimized control of ventilation	Demand-controlled ventilation (DCV)
		MPC
		Other algorithms
	Advanced air distribution	Displacement ventilation
		Underfloor air distribution ventilation
		Personalized ventilation
	Partition	Partitions (semi-open spaces)

		Closing doors between rooms
		Cubicle workstation
		Enclosed/semi-enclosed modular office walls
Air cleaning	In-duct air cleaners	Filtration
		Upper-room UVGI system
	Portable air cleaners	Filtration
		UVGI
		Plasma
		Photocatalytic oxidation (PCO) or UV-PCO
		Sorption
		Plant-based air purifiers
	Other standalone air cleaners	Upper-room UVGI system

Passive air cleaning technologies	Pollutant sorption materials
	Sunlight
PPE	Mask
	Mask fitter/sealer/brace
	Face shield

Table 3-4. IAQ control strategies for mitigating the risk of airborne transmission.

Category	Strategy	Effectiveness	Effective scale	Capital cost <sup>a</sup>	Durability	Accessibility
PPE	Cloth mask	30% particle filtration efficiency for purchased masks (considering leakage from gaps caused by improper fit) (Konda et al. 2020); 20% particle filtration efficiency for homemade masks (cotton and nylon materials) (M. Zhao et al. 2020);	Breathing zone	US\$3/count (purchased); US\$0/count (homemade);	Reusable	Easy <sup>b</sup> ; Homemade (using household materials like cotton and nylon (A. Davies et al. 2013)); Comfort should be considered;
	Surgical mask	50% particle filtration efficiency (considering	Breathing zone	US\$0.2/count	Disposable <sup>c</sup>	Depending on supply capacity (including

	leakage from gaps caused by improper fit) (Konda et al. 2020; Rothamer et al. 2020);				conventional, contingency, and crisis situations (U.S. CDC 2021a)); Comfort should be considered;
N95 mask	95% particle filtration efficiency (without gaps) <sup>d</sup> (Konda et al. 2020)	Breathing zone	US\$2/count	Disposable <sup>c</sup>	Depending on supply capacity (including conventional, contingency, and crisis situations (U.S. CDC 2021a)); Comfort should be considered;
Face shield	23% reduction on aerosol inhalation under long-	Breathing zone	US\$2/count	Reusable	Easy;

		time use (more aerosol reduction under short-distance use) (William G. Lindsley et al. 2014)				Homemade (using plastic sheets and other materials);
Mask fitter/sealer/brace <sup>e</sup>		Greatly improving the efficiency of masks by reducing the leakage through gaps (e.g. improve the particle filtration efficiency of surgical masks to over 90% (Rothamer et al. 2020))	Breathing zone	US\$9/count (purchased (UW Madison 2020)); US\$0/count (homemade);	Reusable	Easy; Homemade (using rubber or other materials (Fix The Mask 2020)); Increased discomfort;
Ventilation of building ventilation	Upgrading filters	99.9% removal efficiency for particles >0.3µm (HEPA filter)	Building	US\$280/count (with an increase in	Replaced every 6-12 months	Easy

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systems (e.g. HEPA)	fan power due to the increased pressure rise through higher-rating filters (Risbeck et al. 2021))
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Increasing outdoor air supply of building ventilation systems (e.g. 100% outdoor air)	Largely increasing the ventilation rate of the room (e.g. providing 60% more clean air compared to a reference system with	Building	US\$5/m <sup>2</sup> ·yea r for extra energy consumption (central system);	Long-term use <sup>f</sup>	Easy (switching the damper to increase outdoor air supply)
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	25% outdoor air and MERV 8 filter)		US\$10/m <sup>2</sup> ·ye ar for extra energy consumption (individual unit) <sup>g</sup> (U.S. EIA 2018);		
Personal ventilation (for fresh air supply)	Increasing clean air supply to the personal microenvironment with typical ventilation effectiveness of 1.7, depending on diffuser location and airflow rate (A. Melikov et al. 2007)	Personal environme nt	Varying greatly (depending on system scale)	Long-term use (filters need regular replacement s)	Need professional design and installation



Local air exhaust	Reducing the cross-contamination with typical ventilation effectiveness of 1.4 to 10, depending on the local partition configuration, exhaust location, and airflow rate (Dygert and Dang 2012)	Personal environment	Varying greatly (depending on system scale)	Long-term use	Need professional design and installation
Displacement ventilation	Improving the clean air delivery efficiency with typical ventilation effectiveness of 1.2 to 2 (Q.Chen and Glicksman 2003)	Room	Varying greatly (depending on system scale)	Long-term use (filters need regular replacement)	Need professional design and installation

Natural ventilation	Providing more outdoor air to the room, e.g. >10h <sup>-1</sup> air changes in some scenarios (H. Qian et al. 2010) (depending on window opening conditions, wind condition, and building geometry) <sup>h</sup>	Room	US\$0	Long-term use	Depending on weather and outdoor air quality conditions
Closing doors between rooms (separating rooms and blocking air flow across rooms)	Reducing cross contamination across rooms (cross-contamination risk can be eliminated when the doors are fully closed)	Building/room	US\$0	Long-term use	Easy (need to coordinate with occupant schedules)

Partition	Partition screens	Depending on airflow patterns in the room (well-designed partitions can reduce the risk of cross-contamination while bad designs may increase the risk in some locations) (Rooney et al. 2021)	Personal environment	US\$100/m <sup>2</sup> material	Long-term use	Easy
Cubicle workstation	Reducing cross-contamination between cubicles with typical ventilation effectiveness of 1.1 to 3.6 (require well-designed ventilation and air distribution) (Haghighat et al. 1996)	Personal environment	US\$2000 for a 2×2m cubicle	Long-term use	Need professional design and installation (incorporating with ventilation)	

	Enclosed/semi-enclosed modular office walls	Enclosed modular office walls can fully avoid cross-contamination between modular spaces (require sufficient ventilation supplied to each space)	Personal environment	US\$3000 for a 2×2m seat	Long-term use	Need professional design and installation (incorporating with ventilation)
Air cleaning and disinfection	Upper-room UVGI system	Proper use can largely improve the inactivation rate of virus (e.g. 16h <sup>-1</sup> for <i>M. parafortuitum</i> ) <sup>i</sup>	Room	US\$1200-2500 (US\$40-90/m <sup>2</sup> floor area) (VirusLights 2021)	Long-term use	Need professional design and installation (incorporating with ventilation)
	Portable air cleaners	Typically supplying 361m <sup>3</sup> /h CADR (median	Room	US\$493/coun t (i.e.	Long-term use (filters	Easy;

	CADR from a market survey (B. Zhao et al. 2020b))		US\$1.32 per m <sup>3</sup> /h CADR, median price from a market survey (B. Zhao et al. 2020b)) (less than US\$50/count for HEPA filter replacement)	need regular replacement )	Potential emissions should be considered (some air cleaners can generate hazardous emissions like ozone (C. Guo et al. 2019))
Sunlight	Increasing the natural inactivation of SARS-CoV-2 on aerosols (decay	Room	US\$0 <sup>j</sup>	Long-term use	Depending on weather conditions and building design <sup>k</sup>

		rate: $<1\text{h}^{-1}$ without sunlight, $>7\text{h}^{-1}$ with low- intensity sunlight, $>18\text{h}^{-1}$ with high-intensity sunlight)				
Occupancy control	Occupancy density restriction (limiting occupant number and social distance in the building/room <sup>m</sup> )	Reducing the potential emissions of pathogens to the air and reducing the risk of cross- contamination	Building/r oom	US\$0 <sup>l</sup>	Long-term use	Easy (need to coordinate with occupant schedules)
	Intermittent occupancy (A. K. Melikov et al. 2020) or	Reducing the potential emissions of pathogens to the air and decreasing the pathogen inhalation by the	Building/r oom	US\$0	Long-term use	Easy (need to coordinate with occupant schedules)

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staggered	susceptible people (e.g.
scheduling (U.S.	asking students to leave
CDC 2021c)	the classroom during the
	15min break after a 35min
	class can reduce 35%
	inhaled pathogen (A. K.
	Melikov et al. 2020))

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<sup>a</sup> Capital costs are approximately estimated based on a brief market survey in the U.S. (data obtained from search results on Google).

The actual costs may vary a lot in other countries or regions.

<sup>b</sup> The strategy can be accessed easily through in-store/online purchase or easy operation.

<sup>c</sup> Surgical and N95 masks are designed for disposable use but can be reused a couple of times after decontamination under certain conditions (U.S. CDC 2021b).

<sup>d</sup> N95 masks can generally fit better than cloth and surgical masks. Proper use of N95 masks can minimize air leakage through gaps.

<sup>e</sup> Other strategies that can maximize fit for surgical masks include 1) covering the surgical mask with a cloth mask, and 2) knotting ear loops of the surgical mask, which can improve the particle filtration efficiency to around 80% (Brooks et al. 2021).

<sup>f</sup> Long-term use indicates a duration of typically over one year.

<sup>g</sup> The annual average cost of air conditioning with central and individual units in the U.S. is US\$0.31 and US\$0.15 per square foot, respectively (U.S. EIA 2018). Assuming the outdoor air fraction of the HVAC system is 25%, applying 100% outdoor air will simply cost 3× more energy for pre-heating or pre-cooling the extra outdoor air.

<sup>h</sup> It can improve the natural ventilation efficiency and introduce more outdoor air to the room incorporating fans.

<sup>i</sup> The effectiveness of virus inactivation by the upper-room UVGI system also relies on room air distribution. Displacement ventilation may reduce the efficiency as it decreases the residence time of the virus in the irradiated zone (Kanaan and Abou Moughlbay 2018).

<sup>j</sup> Introducing more sunlight to the room may impact the heating and cooling energy consumption for the HVAC system.

<sup>k</sup> Building design like window/wall ratio and building geometry can impact the incident sunlight.

<sup>l</sup> The potential economic loss due to occupancy restriction is not considered.

<sup>m</sup> The safe social distance is recommended as 1.6-3.0m considering the aerosol transmission of exhaled large droplets from talking (Sun and Zhai 2020).



### **3.2.4 Selected green building technologies**

The performance of building technologies is dependent on the design features of the technology. In earlier estimations, the design features of technologies were obtained from literature review. However, the information for many strategies, particularly more recent and novel technologies, is insufficient in literature. According to the review on green building technologies, some technologies have shown great potential for improving IAQ and reducing energy consumption. In this study, two novel IAQ control strategies, including needlepoint bipolar ionization technology and displacement ventilation integrated with partitions, that were thought to be effective for mitigating airborne infection risks were studied in detail through chamber experiments and CFD simulations. Because of the enhanced ventilation efficiency, energy use for providing the equivalent ventilation rate by these technologies is lower. Therefore, the energy performance of these technologies was also discussed. The results from this study can contribute to the proposed database, which can be used by more researchers who are interested in these novel technologies.

#### **3.2.4.1 Needlepoint bipolar ionization technology**

Ionization refers to the process of either removing or adding electrons from neutral atoms or molecules, resulting in the creation of ions carrying electric charge. Thus, it has the potential for eliminating particles and bioaerosols (Hyun et al. 2017; Nunayon et al. 2019; Pushpawela et al. 2017; U.S. EPA 2022d). However, the process may also result in the formation and destruction of some volatile organic compounds (VOCs) (Zeng et al. 2021) and other gaseous byproducts such as ozone and nitrogen oxides (NO<sub>x</sub>), particularly at higher ionization voltages (C. Guo et al. 2019). Earlier studies also observed adverse effects on cardiovascular health when people were exposed

to  $13 \times 10^3 \text{ \#/cm}^3$  and  $60 \times 10^3 \text{ \#/cm}^3$  level of airborne negative ions (W. Dong et al. 2019; W. Liu et al. 2021).

Needlepoint bipolar ionization technology is a technology using low ionization voltage to minimize the generation of ozone and ozone-associated secondary emissions but still compromise the pollutant removal capability (Waddell 2019). Some experts have recommended it as a possible mitigation strategy to against COVID-19 (Berry et al. 2022). However, recent studies (Zeng et al. 2021, 2022) indicated the research gap of adequate peer-reviewed evidence on the efficacy and safety of using air ionization technologies to address IAQ issues in practice. In the absence of peer-reviewed evidence, this technology was considered by some public agencies to be “emerging” with little available research outside of lab conditions (U.S. CDC 2020; U.S. EPA 2020).

Therefore, this study performed chamber experiments for two in-duct needlepoint bipolar ionization devices with different configurations in a full-scale office space built up inside the well-controlled environmental chamber. A thermal manikin wearing a pre-soiled shirt was used to mimic a real occupant in the office. Two different testing approaches were conducted, including decay tests and constant-source tests. Realistic room configurations and environmental settings were established during the tests. The performance of pollutant removal (size-speciated particles and VOCs), and potential secondary emissions such as ozone, VOCs, and negative ions were analyzed. The favourable configurations of in-duct ionizations for enhanced pollutant removal and minimal secondary emissions were discussed.

The ionizers were installed in a blower system that was used to simulate the realistic air handling unit (AHU). A regular MERV 8 filter was installed at the outlet of blower system. The operating airflow rate ( $Q_{AC}$ ) of the blower in this study was controlled at 505 m<sup>3</sup>/h (297 CFM, calibrated by the TSI AccuBalance Air Capture Hood 8380, see Supplementary Information). The two ionizers were installed inside the blower, one was a portable unit and installed 0.25 m upstream of the filter (Ionizer B or I-B), and another one was mounted on the MERV 8 filter (Ionizer A or I-A). The blower system and the ionizers are illustrated in Figure 3-2.

Full-scale chamber tests were performed in the Building Energy and Environmental System Laboratory (BEESL) at Syracuse University. A full-scale single office was built inside the full-scale environmental chamber with carpet floor, painted gypsum wallboards for walls and ceiling, a single workstation (a wooden table, a laptop, and partitions), a manikin wearing a pre-soiled T-shirt, and two mixing fans on the corners. The test room was originally built in November 2018 and all interior materials and facilities had been exposed to the laboratory condition for more than two years before the tests. The workstation, manikin, and mixing fans were set up before the presented study (October 2021). The test room has a dimension of 3.5 × 3.0 × 2.7 m (length × width × height) with a room volume of 29.1 m<sup>3</sup>. The air cleaner (blower system) was placed inside the test room (behind the manikin). The environmental control system of the chamber was directly connected to the test room. The supply air entered the test room through a rectangular ceiling diffuser (0.9 × 0.017 m) and was exhausted through a circular outlet with a diameter of 0.1 m close to the floor. The environmental control system can provide a steady stream of fresh air to the test room when needed. The makeup air of the system originated from the outdoor air and filtered by a high-efficiency particulate absorbing (HEPA) filter and an activated carbon filter. The

recirculation damper of the system was completely closed during the test period, so no recirculated air was present.



Figure 3-2. Schematic of the test chamber and the air cleaner (blower system).

Two different test methods and procedures were used in this study, including pollutant decay (or “pull-down”) testing for particles, and constant-source testing for particles and VOCs (Figure 3-3). In the decay tests, particle generation was stopped when the total particle concentration reached about  $1000\text{#/cm}^3$ . A short period of natural decay was performed before turning on the air cleaner. The particle concentration variation was measured during the entire test period (i.e., both before and after turning on the air cleaner) in order to determine the decay rates due to natural deposition

and dilution by any possible infiltration airflow. The air handling unit (AHU) of the chamber system was completely off over the test period, with only natural infiltration presented. The method has been widely used in other studies and standards (ANSI/AHAM 2020).

The constant-source tests were designed to simulate realistic room scenarios with typical indoor particle and VOC levels under a constant ventilation rate. As shown in Figure 3-3, particles and/or VOCs were injected into the testing room continuously throughout the test period. When the background concentration reached a steady state, the blower fan was turned on to remove pollutants through the MERV 8 filter. When another steady state was reached, the ionizer (A or B) was turned on until the final stable state was reached. The steady-state pollutant concentrations at different states were used for estimating the removal efficiency of the air cleaner. The AHU was working continuously during the constant-source tests to provide constant outdoor/fresh air (purified by HEPA and carbon filters) to the indoor air. Two different outdoor airflow rates were provided in this study, including the low airflow ( $Q = 24.0 \text{ m}^3/\text{h}$ ) and the high airflow ( $Q = 55.6 \text{ m}^3/\text{h}$ ). The low airflow ( $Q = 24.0 \text{ m}^3/\text{h}$ ) just met the minimal ventilation required by the ASHRAE standard for office scenarios (i.e.  $20.4 \text{ m}^3/\text{h}$ , based on the area and occupant number of the test room (ASHRAE 2019a)). Thus, the constant-source tests with low ventilation rate can generally represent the worst-case scenario of an office space (high pollutant level and minimal ventilation). The high airflow ( $Q = 55.6 \text{ m}^3/\text{h}$ ) represented the scenarios with improved/enhanced ventilation systems (over  $2\times$  of the minimal requirement). CO<sub>2</sub> decay tests were performed to confirm the ventilation airflow rate of the test room (see Supplementary Information).

A total of 11 tests were performed using different test methods between October 28<sup>th</sup>, 2021, and February 14<sup>th</sup>, 2022, including five decay tests and five constant-source tests (Table 3-5). Decay/pull-down tests were performed to estimate the natural decay rate, the removal rate by MERV 8 filter, the removal rate by ionizer A, and the removal rate by ionizer B. The MERV 8 filter was always utilized when the ionizer A was working because the ionizer was mounted on the filter. For the portable ionizer, ionizer B was tested under two different configurations, with and without MERV 8 filter installed, respectively. The constant-source tests were conducted for testing ionizer A. VOC injection was performed in a constant-source test only (Test 10).

Table 3-5. Test conditions of in-duct NPBI units.

Test method	Test ID	Test condition	Airflow condition	Injected pollutants	Test date
Decay test	Test 1	Natural decay	Infiltration <sup>a</sup>	Particles	Dec 1 <sup>st</sup> , 2021
	Test 2	Blower w/ MERV 8 filter	Infiltration	Particles	Nov 29 <sup>th</sup> , 2021
	Test 3	Ionizer A & filter	Infiltration	Particles	Nov 28 <sup>th</sup> , 2021
	Test 4	Ionizer B & filter	Infiltration	Particles	Nov 30 <sup>th</sup> , 2021
	Test 5	Ionizer B (w/t filter)	Infiltration	Particles	Dec 10 <sup>th</sup> , 2021
Constant-source test	Test 6	Ionizer A & filter	Low airflow <sup>b</sup>	Particles	Oct 28 <sup>th</sup> , 2021
	Test 7	Ionizer A & filter	Low airflow	Particles	Nov 24 <sup>th</sup> , 2021
	Test 8	Ionizer A & filter	High airflow <sup>c</sup>	Particles	Feb 14 <sup>th</sup> , 2022
	Test 9	Ionizer A & filter	High airflow	Particles	Feb 14 <sup>th</sup> , 2022
	Test 10	Ionizer A & filter	Low airflow	VOCs	Nov 23 <sup>rd</sup> , 2021

<sup>a</sup> Infiltration: Environmental system was fully closed. There was only natural infiltration existing in the room,  $\lambda_{inf} = 0.01\text{h}^{-1}$  ( $0.3\text{ m}^3/\text{h}$ ).

<sup>b</sup> Low airflow:  $Q = 24.0 \text{ m}^3/\text{h}$ .

<sup>c</sup> High airflow:  $Q = 55.6 \text{ m}^3/\text{h}$ .



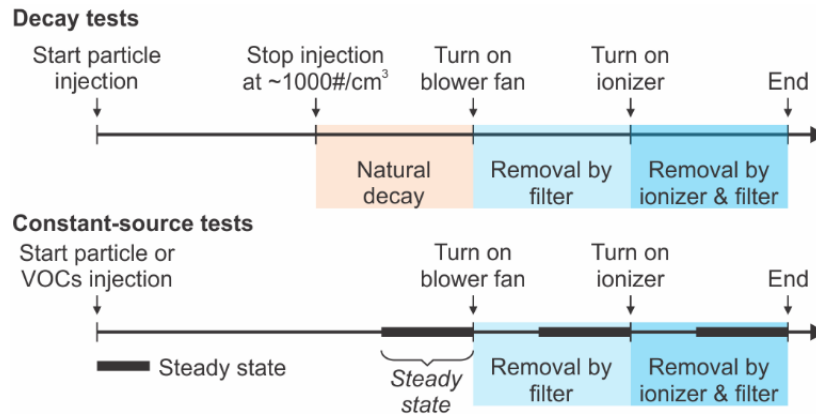


Figure 3-3. Procedures of different test methods.

Particles were disseminated using an 8-jets nebulizer (Blaustein Atomizing Modules (BLAM), CH Technologies Inc) driven by the purified/filtered clean air. The supply air was nominally provided at an airflow rate of 18 L/min with a liquid (salt solution) dissemination rate of approximately 0.14 mL/min. The salt solution was prepared and used for particle generation, which contains 0.05 mol sodium chloride (NaCl) and 0.004 mol anhydrous magnesium sulfate (MgSO<sub>4</sub>) in 0.5 L deionized (DI) water. The solution was injected into the nebulizer by a syringe pump (New Era Pump Systems Inc, Model NE-1000) at the rate of 0.3 mL/min. The generated particles were introduced into the tested room via a vinyl tube with a diameter of 1 cm (3/8 inch). The indoor particle concentrations measured in the presented study were generally within the typical indoor PM<sub>2.5</sub> mass concentration range. In some worst-case scenarios, the indoor PM<sub>2.5</sub> can reach the level of 90 µg/m<sup>3</sup> in offices or 180 µg/m<sup>3</sup> in schools (A. Zhang et al. 2021). The steady-state particle number concentrations monitored in the constant-source tests of this study were below 100 µg/m<sup>3</sup> (Kulkarni et al. 2011; TSI Inc 2012)), which is close to the worst-case indoor PM<sub>2.5</sub> levels. The initial PM<sub>2.5</sub> concentrations of the decay tests were approximately 200 µg/m<sup>3</sup> based on simple

conversions, which were also close to the indoor PM<sub>2.5</sub> levels of the worst-case scenarios. Therefore, the particle generation in this study represented the worst-realistic case scenario of typical indoor particle levels.

In office spaces, over 100 different volatile organic compounds (VOCs) have been identified, although most are presented in relatively low concentrations. Formaldehyde (HCHO) and toluene are the two significant compounds of VOCs in offices. Toluene is often used as a reference compound to quantify the total volatile organic compounds (TVOC). Formaldehyde and Toluene are also the compounds selected to evaluate the performance of various sorbent media in the ASHRAE standard (ASHRAE 2015) and widely used as the target compounds for testing air cleaner performance in other studies (W. Chen et al. 2005; He et al. 2014; Pei et al. 2008; Zhiqiang Wang et al. 2012; X. Wu et al. 2011). They also represent water-soluble and water-insoluble compounds, respectively, representing different challenges faced by air cleaners.

Formaldehyde and toluene were selected as the target compounds to evaluate the removal performance of the tested air cleaning technologies. Constant emission was simulated by using pre-calibrated permeation tubes to achieve a slightly higher but realistic indoor concentration level for testing:

- Formaldehyde: target at a background level of 30 $\pm$ 15 ppb (39.3  $\mu\text{g}/\text{m}^3$ ) with constant injection under the reference condition

- Toluene: target at a background level of 50 $\pm$ 15 ppb (189.6  $\mu\text{g}/\text{m}^3$ ) with constant injection under the reference condition

In order to identify any potential secondary emission released through ozone-initiated reactions with human-related surfaces (e.g. skin or clothes), a soiled T-shirt was put on the thermal manikin as in a previous study (Rai et al. 2014). Prior to the testing day, a freshly cleaned T-shirt was worn by a volunteer overnight. The skin-oil-soiled T-shirt then was put on a heated anatomically correct 20-segment Newton thermal manikin made by Thermetrics (Measurement Technology Northwest Inc.) (Kong et al. 2019). The manikin has a total skin surface of 1.8 m<sup>2</sup> and height of 1.8 m, and was dressed in typical summer clothes, i.e. a T-shirt (soiled) and long trousers (clean), which has a total clothing insulation of 0.43 clo based on the ASHRAE standard (ASHRAE 2017b). The skin temperature was maintained at a set-point temperature of 33.9 °C by a computer-controlled feedback system (Khalifa et al. 2009).

A Dynacalibrator (VICI Metronics, Inc.) was used to supply constant VOC injections to the test room. The clean air (“carrier flow”) went through a temperature-controlled glass chamber, where permeation tubes are located, to carry VOCs permeated from the permeation tubes. The VOC-carrying carrier flow then mixed with the rest part of the clean air (“dilution flow”) to obtain the desired VOC concentration level at the outlet of the Dynacalibrator. The outlet VOC concentration was determined by the airflow rates of carrier flow and dilution flow, and VOC permeation rate from the permeation tube, which is highly dependent on the temperature. The permeation tubes of

formaldehyde and acetone (Dynacal permeation tubes from VICI Metronics, Inc.) were used as the VOC sources of interests in the present study. The Dynacal permeation tubes are small and inert capsules containing a pure chemical compound in a two-phase equilibrium between its gas phase and its liquid or solid phase. At a constant temperature, the device can emit the compound through its permeable portion at a constant rate, which was carried by the airflow over it.

During the experiments, particle concentrations were measured using the TSI Aerodynamic Particle Sizer (APS) 3321 spectrometer, which measures aerodynamic particle size from 0.5 to 20  $\mu\text{m}$ . The total particle concentrations were calculated in the summing mode. Formaldehyde, acetone, and acetaldehyde were collected using 2,4-Dinitrophenylhydrazine (DNPH) sampling tubes connected to the sampling pumps (Alicat Scientific MC-2SLPM-D) with off-line analysis conducted via high-performance liquid chromatography (HPLC). Other VOCs were collected using sorbent tubes connected to the sampling pumps (Alicat Scientific MC-5SLPM-D) and analyzed on an Agilent 6890 gas chromatograph (GC) with an Agilent 5973 mass selective detector (MS). The sampling pumps were calibrated by a flow calibrator (SKC UltraFlo 709), and the total sampling volumes for sorbent and DNPH were 6L and 30L, respectively. The 2B Technologies Model 202 Ozone Monitor was used to measured ozone. Negative ions were monitored using an air ion counter (Alphalab Inc, Model AIC2) with an accuracy of  $\pm 20\%$  of reading. A photoacoustic gas monitor (LumaSense Inc., Innova 1412i) and a CO<sub>2</sub> logger (HOBO Inc., Onset MX1102) were used to monitor the CO<sub>2</sub> concentration decay to determine the airflow rate and the infiltration rate of the room.

All samples were monitored at the exhaust of the room (return air). For decay tests and constant-source tests, particle, ozone, and ion concentrations were continuously monitored throughout the test period. For VOC sampling in Test 10, the sorbent and DNPH samples were collected during each steady state. One set of background samples was taken in the steady state before turning on the blower fan. Two sets of samples were collected in the steady state with the blower turned on, and two other sets were taken in the steady state with the ionizer working. For each set of sampling, two duplicated sorbent samples were collected to obtain duplicated results from GC/MS analyses. Since the DNPH samples can be analyzed repeatedly by HPLC, single sample was collected in each steady state. But each collected DNPH sample was analyzed repeatedly to get two sets of results as for the sorbent samples. A schematic of different test procedures was shown in Figure 3-3. Besides the pollutant samples, the electric energy powers of the tested ionizers were also monitored. A power line meter (Electronic Product Design, Inc.) was used to monitor the electric power when the ionizer was working.

The steady-state indoor pollutant concentration in a well-mixed space can be presented as a simplified single-zone model,

$$\lambda C_{in} = \lambda C_{out} - k_{dep} C_{in} + \frac{E'}{V} \quad (3-1)$$

where  $C_{in}$  is the steady-state indoor pollutant concentration,  $C_{out}$  is the concentration of the inlet air (outdoor air),  $\lambda$  is the total equivalent ventilation rate of the test room,  $k_{dep}$  is the

deposition/removal rate of the pollutant in the room,  $E'$  is the emission rate of the indoor source, and  $V$  is the room volume. For decay tests without any indoor emission sources, the indoor pollutant concentration at time  $t$  ( $C_t$ ) depends on the initial concentration of the pollutant ( $C_0$ ), the total equivalent ventilation rate  $\lambda$  of the room, and the decay time  $t$  (Eq. 3-2). Therefore, exponential regressions can be performed to time-sequential particle and CO<sub>2</sub> concentrations to estimate the particle removal rates and the ventilation/infiltration rates of the room, respectively. For particle decay tests in this study, the estimated removal rate ( $\lambda$ ) included the natural decay rate  $k_n$  (including infiltration  $\lambda_{inf}$  and deposition  $k_{dep}$ ) and the removal rate of the air cleaning unit  $k_{AC}$  (including the removal by filter  $k_{filter}$  and the removal by ionizers  $k_{ionizer}$ ) as Eq. 3-3. For CO<sub>2</sub> decay tests, the estimated ventilation rate ( $\lambda$ ) from exponential expressions can represent the infiltration rate of the room  $\lambda_{inf}$  (without any mechanical ventilation) or the combination of the infiltration rate and the ventilation rate by the control system  $\lambda_Q$  (Eq. 3-4).

$$C_t = C_0 \cdot e^{-\lambda t} \quad (3-2)$$

$$\lambda = k_n + k_{AC} = (\lambda_{inf} + k_{dep}) + k_{filter} + k_{ionizer} \quad (3-3)$$

$$\lambda = \lambda_{inf} + \lambda_Q = \lambda_{inf} + \frac{Q}{V} \quad (3-4)$$

The clean air delivery rate (CADR) is a factor widely used for evaluating the performance of air cleaning technologies. It represents the equivalent “clean” air supplied by the air cleaning

technology for a certain pollutant such as particle or formaldehyde. For decay tests in the present study, the CADR can be determined by the estimated pollutant removal rate ( $k_{AC}$ ) and the room volume ( $V$ ) as Eq. 3-5. For constant-source tests, the CADR of the air cleaner can be derived from Eqs. 3-1 and 3-6. It is dependent on the clean/outdoor air flow rate supplied to the test room by the control system ( $Q$ ), and steady-state pollutant concentrations between the background (without air cleaner,  $C_1$ ) and the stable concentration with air cleaner working ( $C_2$ ), as well as the deposition rate of the pollutant  $k_{dep}$  (assuming a constant deposition rate) (Eq. 3-6). The equivalent single-pass pollutant removal efficiency (SPRE) can be determined by the estimated CADR for a certain pollutant and the airflow rate of the air cleaner ( $Q_{AC}$ ) (Eq. 3-7). The SPRE estimated in this study represented the equivalent removal efficiency of the tested air cleaner for a specific pollutant, including the contributions from all possible removal mechanisms such as infiltration and deposition. It is different from the directly measured single-pass infiltration efficiency based on the concentration difference between downstream and upstream of the air cleaner.

$$CADR = k_{AC} \cdot V \quad (3-5)$$

$$CADR = (Q + k_{dep} \cdot V) \cdot \left( \frac{C_1}{C_2} - 1 \right) \quad (3-6)$$

$$SPRE = \frac{CADR}{Q_{AC}} \quad (3-7)$$

The generation rates of certain compounds or byproducts in the room, e.g., ozone or VOCs, can be calculated through the single-zone model (Eq. 3-1). In this study, the generation rate of ozone was estimated. Indoor ozone originates from the outdoor air and the emission of the air cleaner and removed through deposition on material surfaces and reactions with some gaseous compounds. The overall ozone deposition rate in the test room was estimated in an earlier test performed on May 19<sup>th</sup>, 2021, which was used in this study (assuming constant deposition rate).

The infiltration rate of the test room was approximately  $0.01\text{h}^{-1}$  through CO<sub>2</sub> decay tests ( $R^2 > 0.99$ , see Appendix). All detected indoor particles were below  $2\text{ }\mu\text{m}$  in size with a median diameter roughly between  $0.7$  and  $0.8\text{ }\mu\text{m}$ . The natural decay test (Test 1) illustrated that the natural decay rate of total particle concentrations ( $k_n$ ), that is the combined effect of infiltration ( $\lambda_{inf}$ ) and particle natural deposition ( $k_{dep}$ ), was  $0.22\text{h}^{-1}$  ( $R^2 > 0.99$ ), which also revealed a particle natural deposition rate ( $k_{dep}$ ) of  $0.21\text{h}^{-1}$ . Table 3-6 showed the removal rate, CADR and SPRE for particulate matter (PM) removal of each decay test under the given experimental conditions. The regular MERV 8 filter illustrated a very low  $\text{SPRE}_{\text{PM}}$  under current settings, around 1.7%, which was generally close to the lower bound of the particle removal efficiencies of MERV 8 filters tested by other studies (Alavy and Siegel 2020; Azimi et al. 2014; B. Shi et al. 2012; B. Shi and Ekberg 2015; Stephens and Siegel 2012; Zeng et al. 2022). The combined use of ionizers (A and B) and MERV 8 filter showed an absolute increase on  $\text{SPRE}_{\text{PM}}$  by about 6-7.4%, which was 3.6 to 4.5 times higher than the efficiency of the tested filter. The ionizer B alone (i.e., without the MERV 8 filter) provided very limited particle removal efficiency (3.8%).



In this study, ionization can generally remove particulate matter from the indoor air through two approaches. One is the enhanced deposition of airborne particles to the indoor surfaces due to the aggregation of charged particles (increased particle size) and the attraction of charged particles by the surfaces. The other is the elevated particle filtration/capture by the charged filter as the generated ions attached on the filter. The aggregation of charged particles can also be filtered more readily attributed to the increased particle size. According to the collected ion concentrations (Section 3.2), most ions generated by the ionizer were trapped/captured by the filter when the filter was installed at the downstream of the ionizer. Therefore, the particle removal of using ionizer B individually ( $CADR_{PM}$  of  $19.4 \text{ m}^3/\text{h}$  and  $SPRE_{PM}$  of 3.8% in Test 5) was merely attribute to the enhanced deposition of charged particles to the indoor surfaces, which also represented the maximum particle removal contribution through enhanced deposition. Considering the notable drop of ion concentrations through the filter, the actual particle removal due to the enhanced deposition in the air may contribute less when the filter is installed and integrated with the ionizer. According to the test results, the integrated use of ionizer B and filter (Test 4) almost doubled the  $CADR_{PM}$  and equivalent  $SPRE_{PM}$  compared to ionizer B alone (Test 5). Based on the analysis above, the enhanced filtration/capture by the charged filter likely dominated the particle removal for the integrated use of ionizer and filter, while the enhanced deposition from the air to indoor surfaces had less effects.

Table 3-6. Performance of different units during chamber tests.

Tests	Test 1	Test 2	Test 3	Test 4	Test 5
Test conditions	Natural decay	Filter (MERV 8)	I-A + filter	I-B + filter	I-B (w/t filter)
Natural decay rate <sup>a</sup> [h <sup>-1</sup> ]	0.22	0.25	0.24	0.25	0.20
Removal rate [h <sup>-1</sup> ]	/	0.54	1.82	1.59	0.87
		<b>0.29</b> (w/t decay)	<b>1.58</b> (w/t decay)	<b>1.34</b> (w/t decay)	<b>0.67</b> (w/t decay)
CADR <sub>PM</sub> [m <sup>3</sup> /h]	/	15.7	53.0	46.3	25.3
		<b>8.5</b> (w/t decay)	<b>46.0</b> (w/t decay)	<b>38.9</b> (w/t decay)	<b>19.4</b> (w/t decay)
SPRE <sub>PM</sub> [%]	/	3.1	10.5	9.2	5.0
		<b>1.7</b> (w/t decay)	<b>9.1</b> (w/t decay)	<b>7.7</b> (w/t decay)	<b>3.8</b> (w/t decay)

<sup>a</sup> Natural decay rate ( $k_n$ ): the combined effect of infiltration ( $\lambda_{inf}$ ) and particle natural deposition ( $k_{dep}$ ).

The results of decay tests were generally consistent with the results of constant-source tests. Figure 3-4 showed the steady-state total particle concentration before and after using the ionizer A in four constant-source tests. The 1-h stable data of the background, the state with only MERV 8 filter, and the state with the ionizer A on, were selected and calculated. The background concentrations in Tests 6 and 7 did not reach a steady state due to the large injection fluctuations during the tests, and thus, were not presented in this study. An example of time-sequential total particle concentration distribution during the test period (Test 9) was presented in Figure 3-4 (b). After the ionizer A was turned on, the total particle concentrations sharply declined. As shown in Table 3-7, the blower with the MERV 8 filter under the given conditions (Tests 8 and 9) provided 4.9-8.3 m<sup>3</sup>/h CADR<sub>PM</sub>, which resulted in SPRE<sub>PM</sub> between 1.0% and 1.6%, consistent with the results of decay tests. The particle natural deposition rate  $k_{dep}$  (0.21h<sup>-1</sup>) measured in decay tests was applied in Eq. 3-6 for estimation (assuming constant particle natural deposition). The actual particle deposition rates (when the ionizers were working) were probably underestimated. They could be higher than the natural deposition rate estimated without the ionizers. In this case, the CADR<sub>PM</sub> and SPRE<sub>PM</sub> estimated through constant-source tests might be underestimated as well. The estimated CADR<sub>PM</sub> of the integrated uses of ionizer A and filter was 36.5-45.1 m<sup>3</sup>/h and the SPRE<sub>PM</sub> was 7.2-8.1%. It is also consistent with the CADR<sub>PM</sub> and SPRE<sub>PM</sub> estimated through the decay test for ionizer A (Test 3). The steady-state particle concentration variations before and after using the ionizer also indicated the particle removal performance of ionizer A. The estimated CADR<sub>PM</sub> provided by the ionizer A individually were between 24.8 and 68.5 m<sup>3</sup>/h, which is 1.3× to 3.4× of the minimal ventilation requirement per person (ASHRAE 2019a), and the estimated SPRE<sub>PM</sub> of the ionizer were around 4.9-13.5% under the given settings (Tests 6-9). During the constant-source test Test 8, a particle concentration decay test was also performed. The overall

decay rate estimated from the exponential regression was  $3.86 \text{ h}^{-1}$  ( $R^2 > 0.99$ , including natural deposition and dilution by the fresh air supplied by the control system Q). Then, the  $\text{CADR}_{\text{PM}}$  supplied by the ionizer A and filter was  $52.6 \text{ m}^3/\text{h}$  (excluding particle deposition and dilution by the fresh air supply), assuming a constant particle deposition rate of  $0.21 \text{ h}^{-1}$ . The  $\text{SPRE}_{\text{PM}}$  by the ionizer and filter was 10.4%.

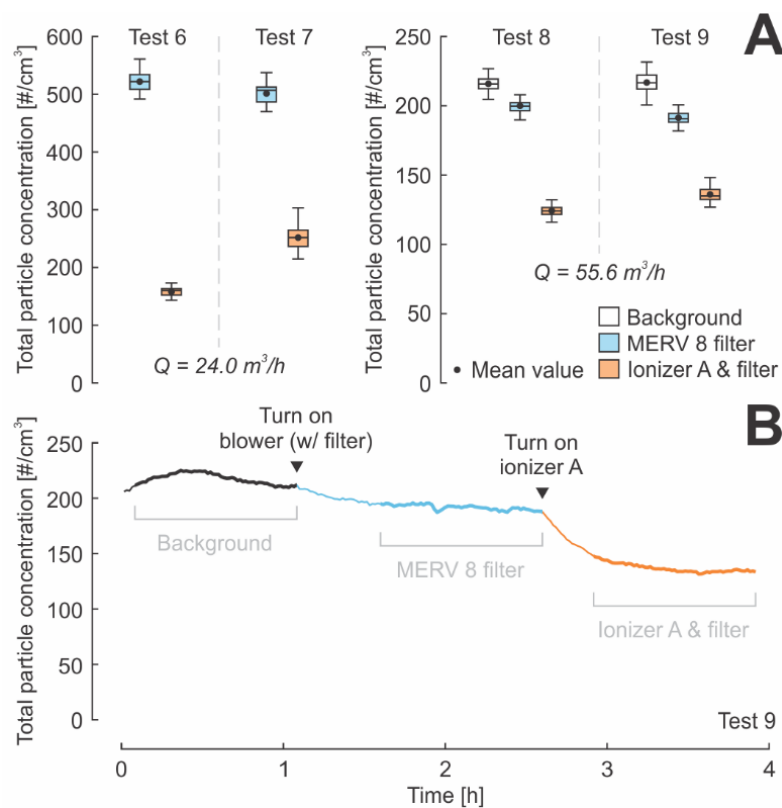


Figure 3-4. (a) Total particle concentration distributions of different states in constant-source tests; (b) An example (Test 9) of time-sequential total particle concentration distribution during the test period.

Table 3-7. Particle removal performance of constant-source tests.

Test	Q [m <sup>3</sup> /h]	$k_{dep} \cdot V^a$ [m <sup>3</sup> /h]	Average total particle number level in each state [# / cm <sup>3</sup> ]			CADR <sub>PM</sub> <sup>b</sup> [m <sup>3</sup> /h]			SPRE <sub>PM</sub> <sup>b</sup> [%]		
			Bkg	Blower on (w/ filter)	Ionizer A on	MERV 8 filter	Ionizer A & filter	Ionizer A <sup>c</sup>	MERV 8 filter	Ionizer A & filter	Ionizer A
6	24.0	6.1	/	521.4	159.1	/	/	68.5	/	/	13.6
7	24.0	6.1	/	500.7	251.7	/	/	29.8	/	/	5.9
8	55.6	6.1	215. 5	199.7	124.5	4.9	45.1	37.3	1.0	8.9	7.4
9	55.6	6.1	217. 3	191.6	136.6	8.3	36.5	24.8	1.6	7.2	4.9
8 <sup>d</sup>	55.6	6.1	/	/	/	/	52.6 <sup>e</sup>	/	/	10.4 <sup>e</sup>	/

<sup>a</sup> Assuming a constant particle deposition rate  $k_{dep} = 0.21 \text{ h}^{-1}$ . The estimated  $\text{CADR}_{\text{PM}}$  and  $\text{SPRE}_{\text{PM}}$  for the scenario with ionizer A and filter, and the scenario with ionizer A individually, may be underestimated as the actual particle deposition rate was likely higher when the ionizer was working.

<sup>b</sup> Particle removal due to natural deposition were ruled out.

<sup>c</sup> The CADR of ionizer was calculated through Eq. 3-6, instead of subtracting the CADR of filter from the CADR of ionizer and filter.

<sup>d</sup> A decay test was performed along with the constant-source test of Test 8. The overall decay rate estimated from the exponential regression was  $3.86 \text{ h}^{-1}$  (including natural deposition and dilution by fresh air supplied by the control system Q).

<sup>e</sup> Excluding natural deposition and dilution by fresh air supply.

When the filter was installed at the downstream of the ionizer, a considerable number of produced ions were likely captured by the filter. The steady-state negative ion concentrations were only  $1.1 \times 10^3 \text{ \#/cm}^3$  and  $1.5 \times 10^3 \text{ \#/cm}^3$  for ionizer A and B of the decay tests, respectively. For constant-source tests, the indoor negative ion levels were around  $0.9 \times 10^3$  to  $1.4 \times 10^3 \text{ \#/cm}^3$  in the air for ionizer A. The measured negative ion concentrations were still close to the background level. However, if the filter was removed, most ions were introduced to the indoor air and the indoor negative ion level increased dramatically, i.e. approximately  $16.1 \times 10^3 \text{ \#/cm}^3$  for ionizer B in the scenario without the filter, over 10 times higher than the level with the filter installed. The result indicated a significant reduction of around 91-94% through the filter. A previous study observed an almost 100% drop of air ions through higher-efficiency MERV 11-12 filters, although the downstream ion concentrations were measured at a longer distance after the filter, at which distance, many air ions were likely decomposed on the metal air duct (B. Shi and Ekberg 2015). Therefore, when a filter was installed at the downstream of the ionizer, most of the produced ions were captured by the filter, particularly for higher-efficiency filters.

In decay tests, ozone concentration increases were detected in the confined space after the ionizer was turned on due to zero outdoor ventilation airflow, though the amount of increase was less than 3 ppb in average. The background levels (Bkg) presented the steady-state concentrations over 60 min before the ionizers were turned on. The ionization levels showed the 60-min steady-state data of ozone concentrations when the ionizers were turned on. Significant ozone concentration increases were observed during the ionization periods ( $p\text{-value} < 0.001$  for three tests), but the increases were fairly small. As carbon filters were used in HVAC systems of the chamber and the laboratory, the background ozone concentrations in the chamber were almost zero (0-0.1 ppb). The

negative values were within the detection errors. The mean steady-state ozone concentrations of the ionization periods in three tests (ionizer A, ionizer B, and ionizer B without filter) were 1.2, 2.0, and 2.9 ppb, respectively. Our earlier ozone decay test (in Appendix) illustrated that the ozone deposition rate in the same room setting was around  $5.5 \text{ h}^{-1}$  (most ozone deposition occurred on building surfaces and human-related surfaces (Nazaroff and Weschler 2021; Shen and Gao 2018; Yao and Zhao 2018)). Therefore, it can be estimated that the ozone generation rates during the ionization in three tests were 0.39, 0.63, and 0.92 mg/h for ionizer A (with filter), ionizer B (with filter), and ionizer B (without filter), respectively. Relative to other indoor air cleaning technologies that may generate ozone (C. Guo et al. 2019), the ozone emission rates of the tested ionizers were very low. Therefore, under scenarios with realistic room and ventilation settings, the ozone increases by ionization were likely negligible for the tested ionizers, which was closely consistent with the observations in the constant-source tests. No significant ozone increases were detected when certain ventilations were provided to the room (Tests 6-10). It was also observed by other studies and claimed by some needlepoint bipolar ionizer manufacturers (Carrier Global 2021; Waddell 2019; Zeng et al. 2021, 2022).

Indoor VOCs were collected and analyzed in Test 10. The background sampling set (two duplicated samples) were collected at time  $t = 0\text{h}$ . The background formaldehyde and toluene concentrations were around 20.6 and  $138.7 \mu\text{g}/\text{m}^3$ , respectively, which were relatively lower than the target level (39.3 and  $189.6 \mu\text{g}/\text{m}^3$ ) but still within the typical indoor concentration range. Other detected VOCs, such as acetaldehyde, acetone, benzene, and other aldehydes, were relatively lower. Most VOCs were below the level of  $2 \mu\text{g}/\text{m}^3$  (see Appendix). After the blower was turned on and a steady state of indoor VOCs was reached, VOC samples were collected at  $t = 9\text{h}$  and  $t =$



12h (for each set of sampling, two duplicated samples were collected). The mean values of two sets (four samples in total) were calculated. After a certain period since the ionizer (ionizer A) was turned on ( $t = 18\text{h}$  and  $t = 50\text{h}$ ), another two sets of VOC sampling were performed (two duplicated samples for each sampling set were collected).

Reference background VOC levels were measured through a series of GC/MS analyses in the test room with same configurations, except the data of formaldehyde and toluene (injected compounds), which were collected through the real-time proton-transfer-reaction mass spectrometer (PTR-MS; Ionicon Analytik Ges.m.b.H., Austria) sampling in two tests with same injection procedure and test conditions (S. Zhou et al. 2020). Mean, standard deviation (SD), and relative standard deviation of reference background levels were calculated, representing the background fluctuations of VOCs of concern. A few compounds were not detected (n.d.) in some tests, in which case their concentrations were assumed to be zero for calculating mean and SD values. Another two injection tests were performed to study the injection fluctuations of formaldehyde and toluene with the same injection procedure and test settings ( $Q$ : 20.7-22.9  $\text{m}^3/\text{h}$ ).

The results showed that the VOC concentrations under the conditions with blower/filter and ionizer were very close, and both were close to the background level before using the blower. The concentration variations of all VOCs were within the 2 relative SD % of the background levels, which represented the range with about 95% of samples in the same group. Therefore, it is believed in this study that the measured VOC variations are all within background fluctuations or injection fluctuations under the given settings. The TVOC level increased 5% after turning on the ionizer,

which was also a slight variation. Thus, under the test conditions in this study, there were no significant chemical reactions with detected VOCs throughout the ionization process. No significant secondary VOC emissions were generated, and no specific VOCs were removed by the ionizer.

The electric powers of the tested ionizers were very stable when they were working. The average power of ionizer A was 0.4 W, and the average power of ionizer B was 1.0 W. They were significantly lower than the typical power of standalone air cleaners (median: 56 W (B. Zhao et al. 2020b)). Based on the results from decay tests and constant-source tests, the ionizer A can approximately provide an equivalent  $CADR_{PM}$  of 24.8-68.5  $m^3/h$ , while the ionizer B can roughly provide 19.4-30.4  $m^3/h$   $CADR_{PM}$  without considering particle removal by the filter, under the settings of this study. Then the electric power per  $CADR_{PM}$  was 0.006-0.016  $W/(m^3/h)$  and 0.033-0.052  $W/(m^3/h)$  for ionizer A and B, respectively. Therefore, the tested in-duct ionizers were effective for air cleaning/particle removal with low energy cost compared to typical standalone air cleaners with a median electric power per  $CADR_{PM}$  of 0.15  $W/(m^3/h)$  (B. Zhao et al. 2020b). Therefore, the performance of the tested ionizers can be concluded in Table 3-8.

Table 3-8. Performance summary of studied ionizers.

Technology	In-duct needlepoint bipolar ionizer A		In-duct needlepoint bipolar ionizer B	
Description	4 bars mounted on filter (34 +/- poles in total)		Portable unit with 6 pairs of +/- poles	
Design features	CADR <sub>PM</sub> [m <sup>3</sup> /h]	24.8 – 68.5	19.4 – 30.4	
	CADR <sub>VOC</sub> [m <sup>3</sup> /h]	0	0	
	CADR <sub>O<sub>3</sub></sub> [m <sup>3</sup> /h]	0	0	
	SPRE <sub>PM</sub> [%]	4.9 – 13.6	3.8 – 6	
	SPRE <sub>VOC</sub> [%]	0	0	
	SPRE <sub>O<sub>3</sub></sub> [%]	0	0	
	ER <sub>PM</sub> [µg/h]	0	0	
	ER <sub>VOC</sub> [µg/h]	0	0	
	ER <sub>O<sub>3</sub></sub> [µg/h]	390	630 - 920	

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Electric power [W]    0.4

1.0

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### **3.2.4.2 Displacement ventilation integrated with partitions in offices**

Displacement ventilation can typically improve the clean air delivery efficiency with a ventilation effectiveness of 1.2 to 2 comparing to mixing ventilation (Q.Chen and Glicksman 2003). Partitions have been widely used in office settings for creating private spaces. It has also been applied to reduce the transmission of SARS-CoV-2 through droplets during this pandemic when social distance cannot be maintained. Cubicle workstations (Haghighat et al. 1996), enclosed or semi-enclosed modular office wall assemblies (PoppinSpaces 2021), and portable partition screens (Rooney et al. 2021) are the major types of partitions used in office settings. Some studies suggested that the combination of displacement ventilation and partitions can effectively mitigate airborne transmission of respiratory diseases like COVID-19 and reduce cross contaminations (Shen, Kong, Dong, et al. 2021a; J. Zhang 2020). But the actual effectiveness of this strategy has not been tested yet. Therefore, the effectiveness of combined displacement ventilation and partitions in office settings for reducing airborne transmission of respiratory diseases like COVID-19 is investigated in this study. The best design for mitigating infection risk through airborne transmission is discussed when conducting displacement ventilation and partitions in offices. Experiments were conducted in the full-scale office space inside the environmental chamber in BEESL. CFD simulations were also performed to understand more details about the air distribution under certain ventilation settings.

The chamber experiment was performed in the same full-scale chamber with same office settings for testing the ionizers. But in this experiment, two identical tables (1.83×0.60 m with a height of 0.71 m) were placed in the middle of the tested room. Two full-scale manikins were sitting face to face. A semi-circular displacement diffuser (0.41 m in diameter, 0.81 m in height) was placed

attached to one lateral side of the room and delivered 208 m<sup>3</sup>/h fresh air at around 15°C to the room. The displacement ventilation supply air settings were determined by the guidelines (Q.Chen and Glicksman 2003). The surface temperatures of manikins were set to 34°C with long-sleeve shirts and trousers on manikins. A laptop was placed in front of each manikin, while overhead ceiling lights were on during the tests. Two different respiratory conditions were tested, including breathing and coughing. The respiratory settings (pollutant injection settings) are presented in Table 3-11. Four different partition settings were studied (Figure 3-5), including no partition, single-panel partition (Partition A), semi-wrapped partitions (Partition B), and fully wrapped partitions (Partition C). There was a 0.30 m gap between the partition boards and the floor to introduce the supply air from the diffuser. TSI large particle generator was used to generate aerosols to and introduced to the room through the tube attached to the breathing zone of one manikin (i.e. the “virus carrier”). The other manikin was considered as the “susceptible subject”. Airflow velocity and temperature were measured by the probes of the air distribution system. Particle number concentration and size distributions were sampled by the TSI aerodynamic particle sizer (APS) in multiple locations. Figure 3-6 shows the settings for the chamber tests and Figure 3-7 illustrates the sampling locations.

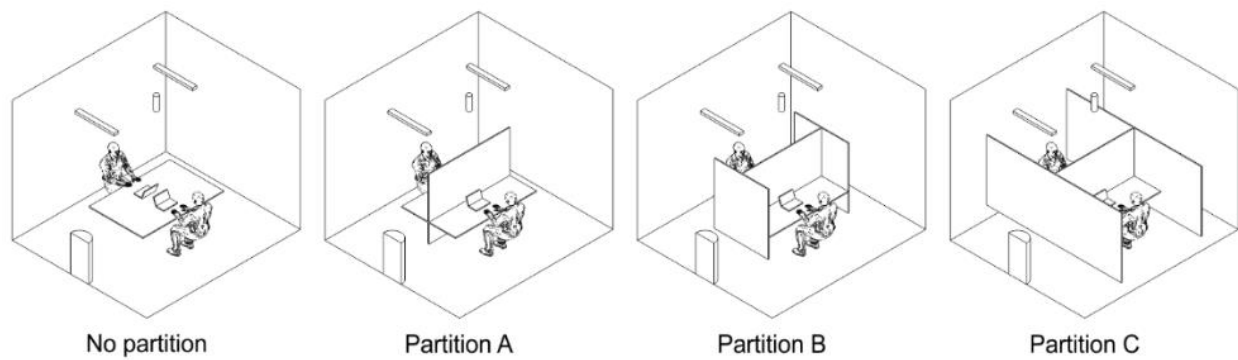


Figure 3-5. Indoor settings of the tested reference office room.

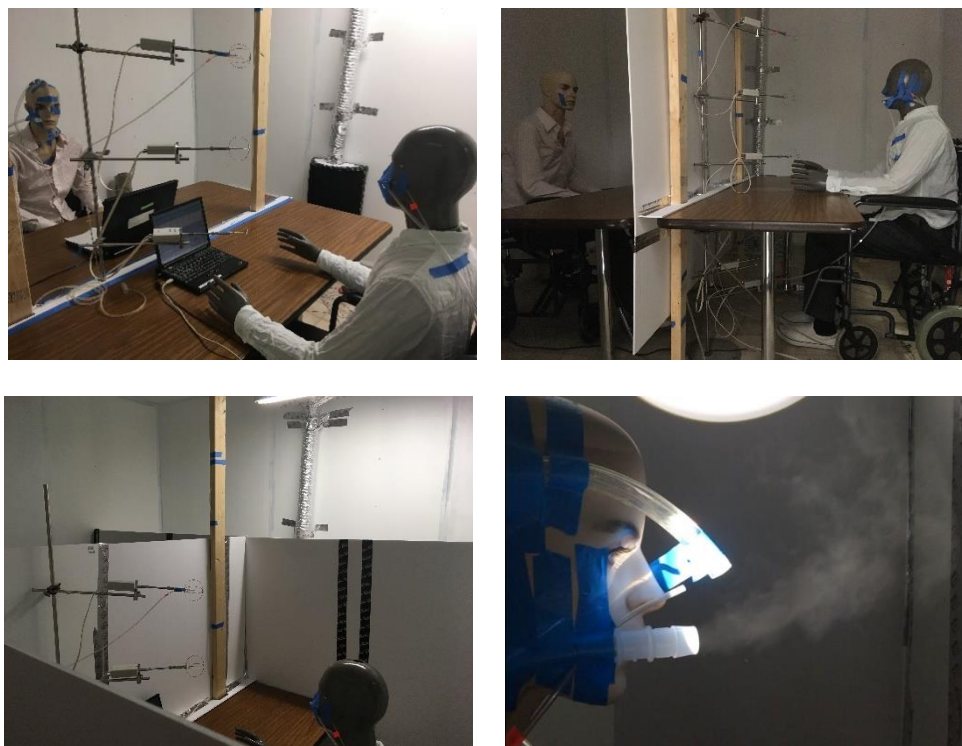


Figure 3-6. Experimental settings of the full-scale chamber tests.

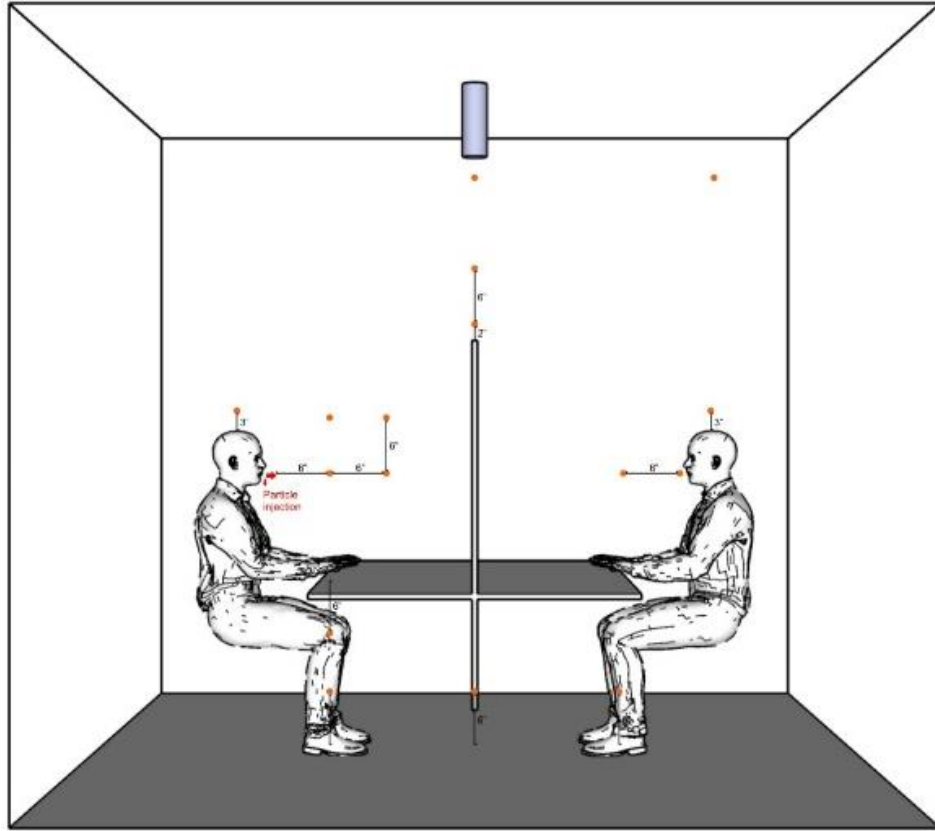


Figure 3-7. Sampling locations.

CFD simulation for the proposed 4 settings were conducted using ANSYS Fluent. Figure 3-8 shows an example of the geometry and mesh settings in CFD simulations. Based on a review on literature (Ahmadzadeh et al. 2021; Assaad et al. 2018; Katramiz et al. 2021; Lai and Cheng 2007; C. Wu and Ahmed 2012; Y. Yan et al. 2020), RNG K-epsilon was used with enhanced wall treatment as the meshes near surfaces are very fine ( $y^+ \leq 1$ ). Full buoyancy effect was applied to study the thermal buoyancy and thermal plume with Boussinesq approximation. SIMPLE scheme was used with second-order accuracies. Discrete phase model (DPM) was used to simulate the particles exhaled by breathing and coughing. CFD settings were presented in Table 3-9.



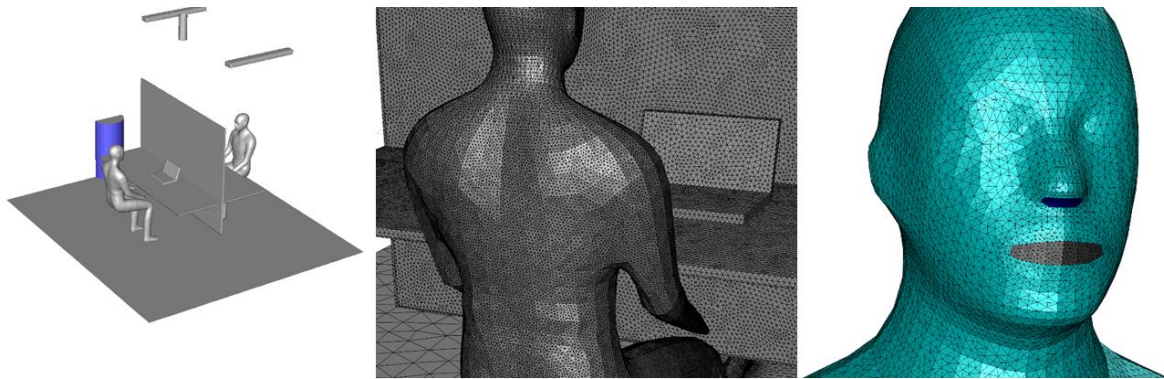


Figure 3-8. Preview of CFD grids.

Table 3-9. CFD settings of the studied displacement ventilation cases.

Facility	Settings
<b>Diffuser</b> (semi-circular diffuser):	Height: 32 in
	Diameter: 16 in
	Air flow rate: 0.049 m <sup>3</sup> /s (~104cfm)
<b>Manikin:</b>	Surface temperature: 33.89 °C
	Total area: 1.81 m <sup>2</sup>
	Surface heat flux (steady-state, total): 70
	W/m <sup>2</sup> (we can assume 40% of total heat flux as convective heat flux, i.e. 28 W/m <sup>2</sup> )

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**Particle injection jet (through round tubes):****Coughing/talking:**

Tube diameter (circle): 5/8 in (0.015875 m)

→ area: 1.98 cm<sup>2</sup>

Injection velocity (uniform): 5.26 m/s

Injection direction: horizontally forward

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**Breathing:**

Tube diameter (circle): 3/8 in (0.009525 m)

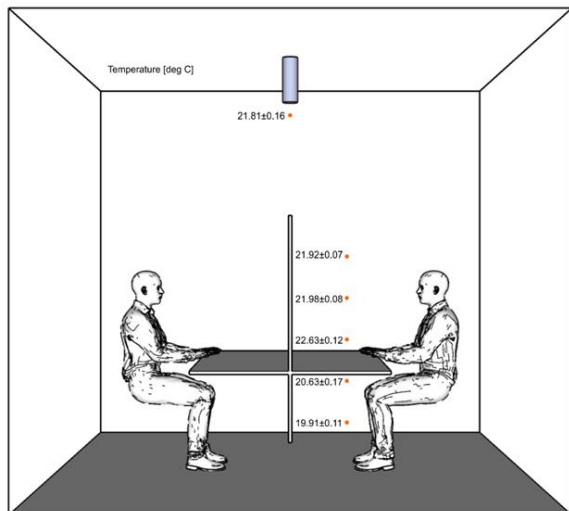
→ area: 0.71 cm<sup>2</sup>

Injection velocity (uniform): 1.56 m/s

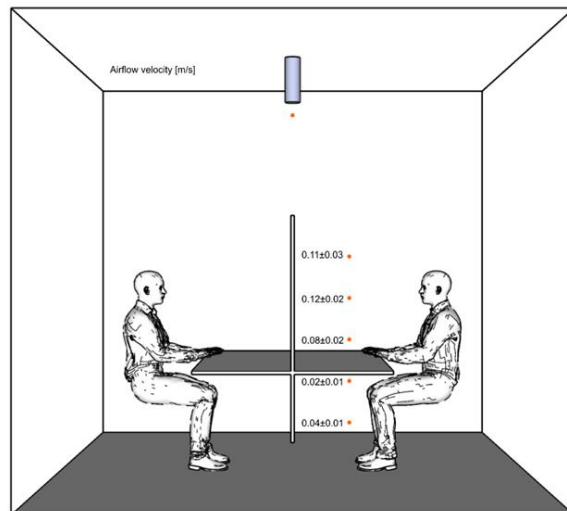
Injection direction: vertically downward

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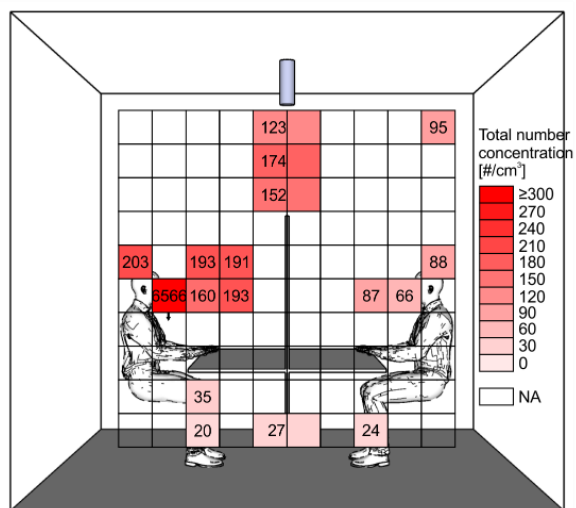
Thermal stratification was observed in the tested room with testing settings. Thermal plume created uprising airflows around manikins and transported abundant quantities of expelled aerosols from the breathing zone of the virus carrier to the exhaust, while only few aerosols were transported to the breathing zone of the susceptible subject (Figure 3-9). Figure 3-10 shows the particle size distribution in the breathing zone of the susceptible subject (receiver).



(a)



(b)



(c)

Figure 3-9. Measured total number concentration in the Partition-A scenario.

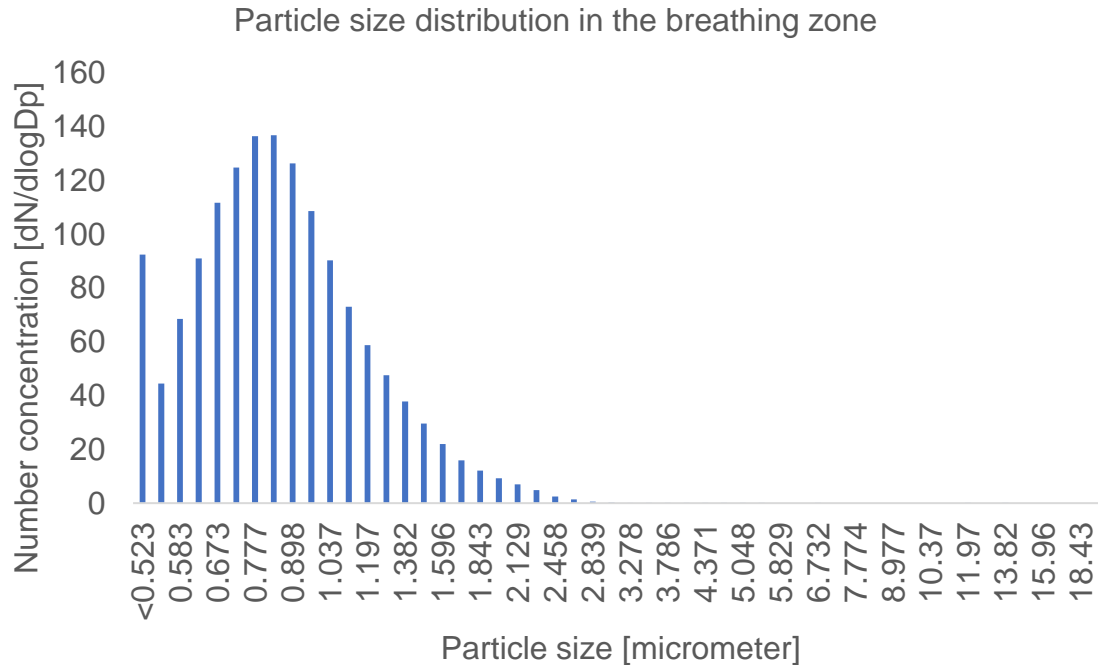


Figure 3-10. Particle size distribution.

The ratios of particle concentration in the breathing zone of the receiver and the particle concentration in the exhaust of the room indicate that displacement ventilation provided higher air distribution efficiency compared to the mixing ventilation (ratio < 1). Under the coughing scenario, adding partitions can further decrease the cross-contamination between the infector and the receiver. Under current settings, the Partition-B can provide highest air distribution efficiency (lowest breathing-zone/exhaust concentration ratio). Adding more partitions to form semi-open spaces does not further block the particle transmission. This may need further studies in future.

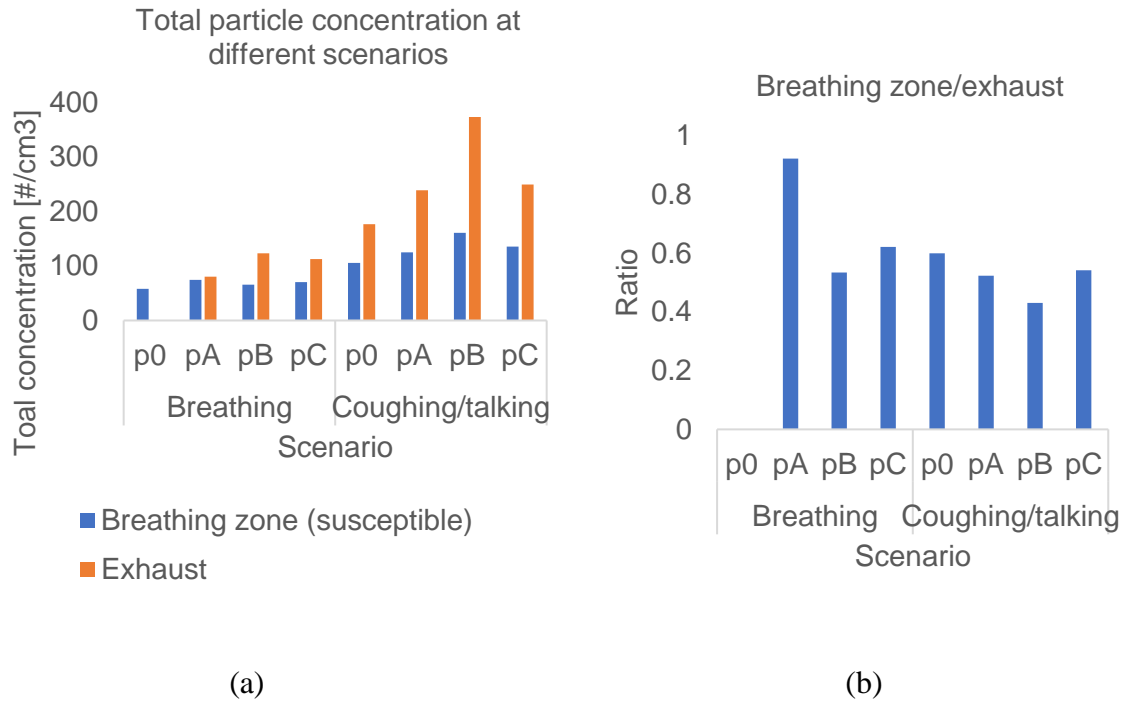


Figure 3-11. Particle concentration at the breathing zone of the receiver and the exhaust of the room.

CFD simulations illustrate similar results. Under talking and coughing scenarios, if partitions were not installed, a huge amount of exhaled particles would be transmitted to the breathing zone of the receiver. However, if partitions were installed and used properly, most particles would be blocked by partitions and exhausted by the exhaust air. Results show that nearly 70% of exhaled particle were deposit on partitions and other surfaces, or removed by the exhaust air. However, adding more partitions does not further dramatically remove more particles, particularly from Partition-B to Partition-C, which is close to the experimental results. It's likely because the dominated airflow in the space the thermal plume surrounding the occupants. Particles were less likely to travel across the partitions and reach the receiver's breathing zone from lateral or behind directions. Therefore,

for workstations in the office scenario, Partitions that cover the workstation may be sufficient to mitigate the infection transmission between people who are sitting face to face. In the mixing ventilation scenario, because of the cross airflow between cubicles, semi-open space can be more effective in reducing cross-contamination.

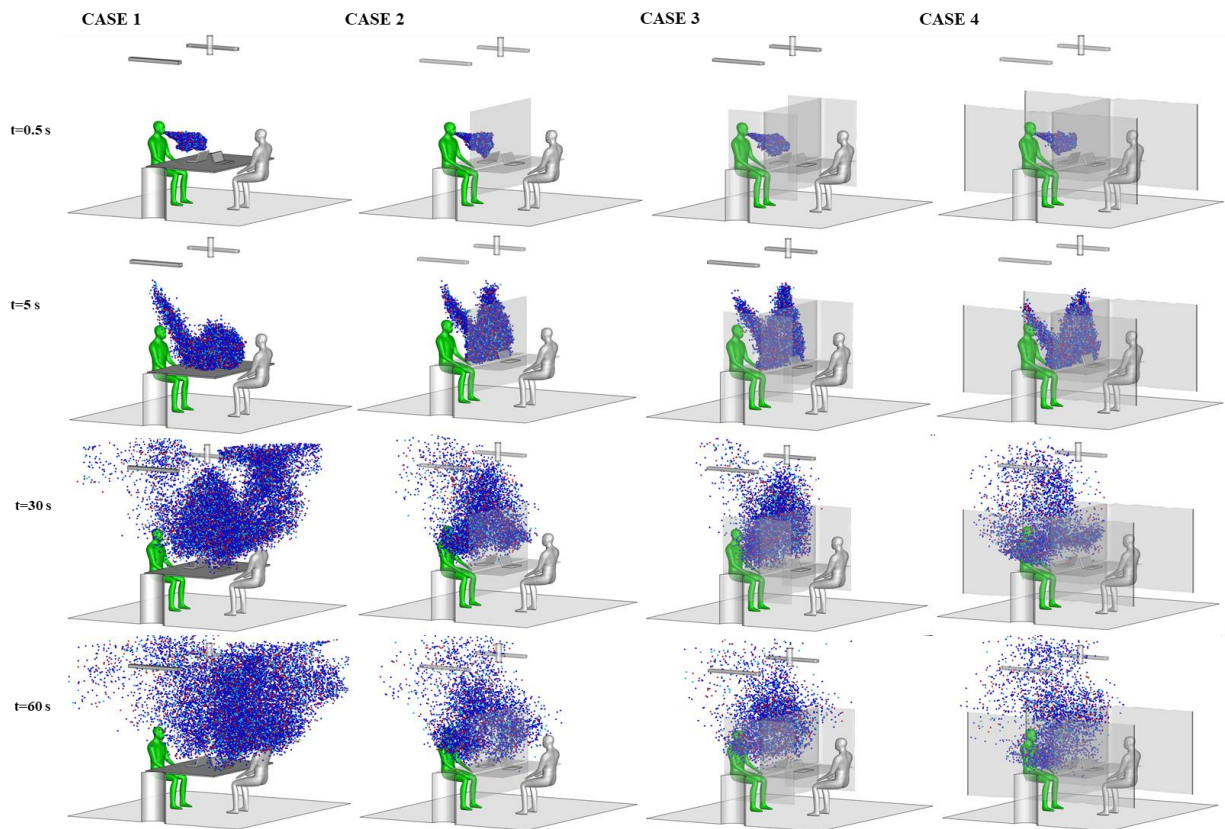


Figure 3-12. Particle distribution over time under coughing scenario.

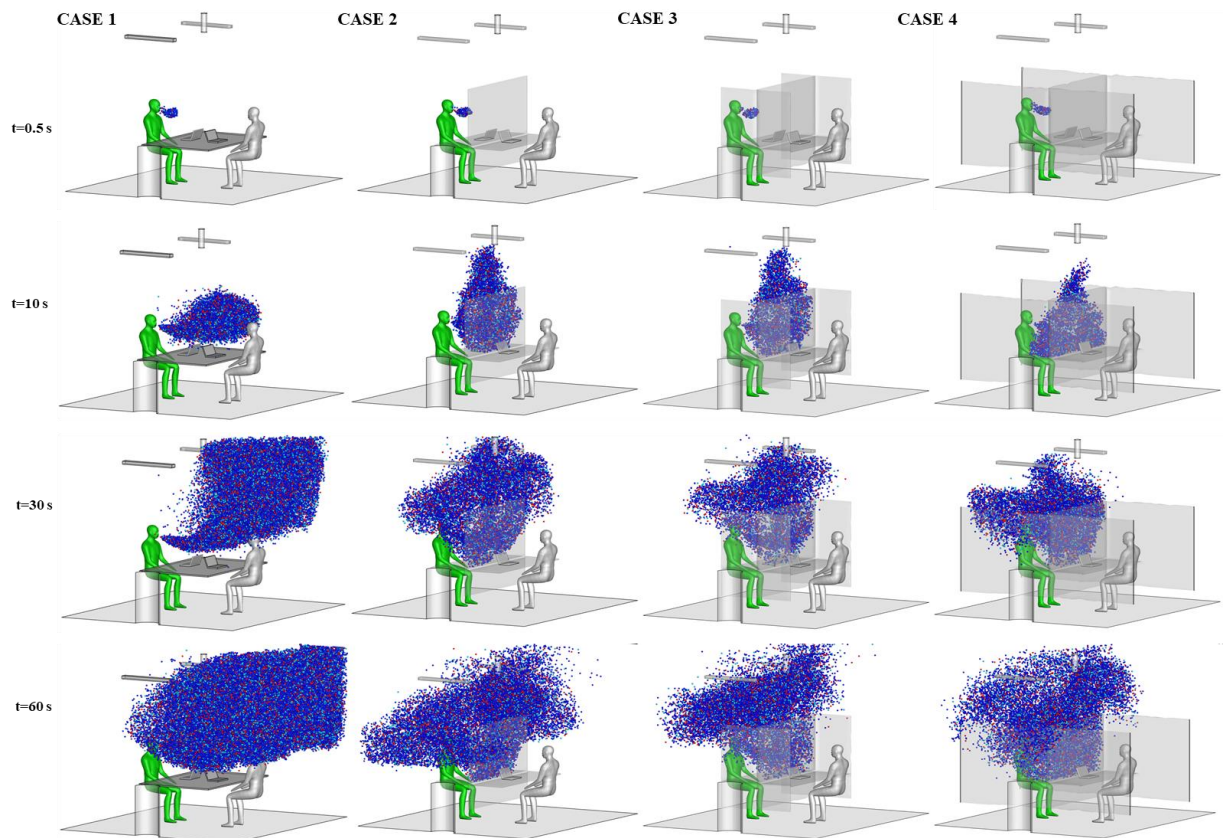


Figure 3-13. Particle distribution over time under talking scenario.

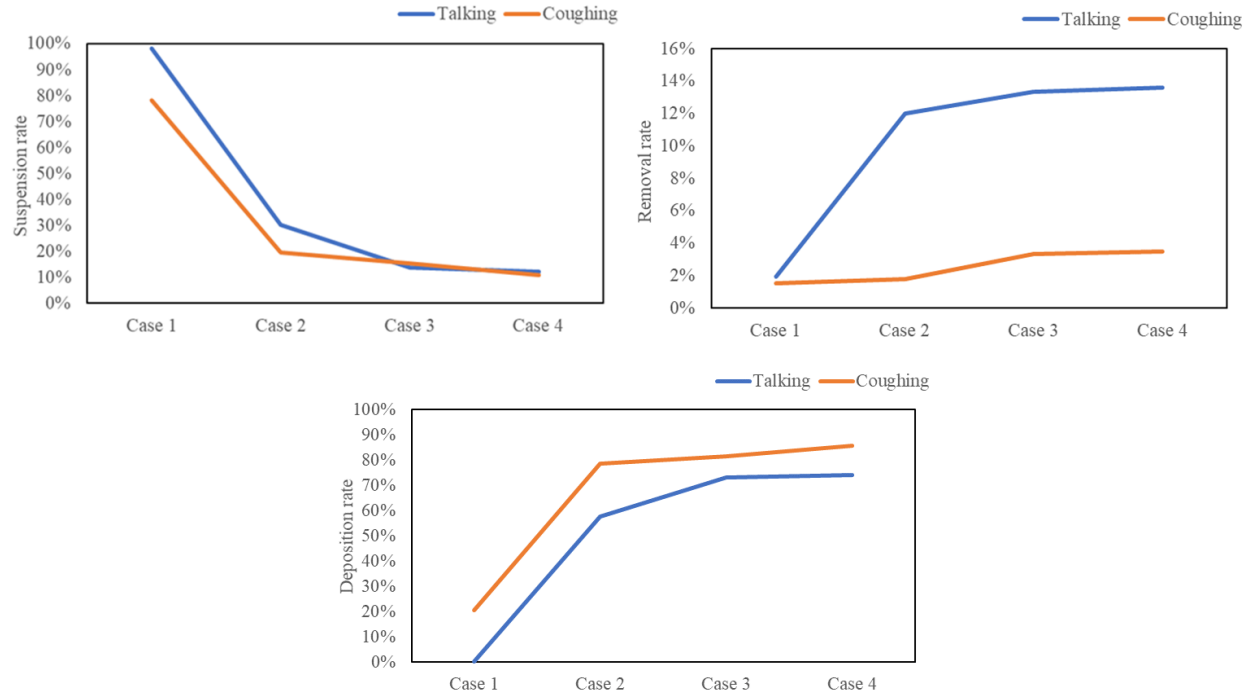


Figure 3-14. Effectiveness of particle removal in different scenarios.

### 3.3 Conclusions

This Chapter introduced a database of typical green building strategies on energy and IAQ improvement. Energy-efficiency strategies include passive technologies such as advanced envelope solutions, improved building service systems (e.g., HVAC, DHW, lighting, and other appliances), and renewable energy generation technologies. Major IAQ improvement strategies include source control, ventilation, and air cleaning. The performance of some strategies was estimated in Chapter 2. IAQ strategies were further discussed as they are directly related to people's health.



Two emerging IAQ technologies were selected and performed chamber experiments and/or CFD simulations, i.e., in-duct needlepoint bipolar ionization (NPBI) units, and the combined use of displacement ventilation and partitions. The in-duct NPBI system can provide an equivalent SPRE of 3.8-13.6% on particle removal without significant ozone and VOCs removal and generation. However, the technology consumes very minor energy consumption, which can result in considerable effectiveness on particle removal and infection risk mitigation with minimal energy use. The combined application of displacement ventilation and desk partitions can effectively mitigate the potential virus transmission between the infector and the receiver through coughing or talking. But adding additional partitions is less effective on mitigating infection risk through breathing as transmission due to breathing is less likely to occur with displacement ventilation. These experiments and CFD simulations provide abundant performance data for the studied technologies. The technologies and their performance metrics will be compiled into the database for green building technologies and systems.

#### **4. Summary and conclusions**

This dissertation developed a modular-based approach for designing buildings with improved performance at early design phase that can provide fast and reliable response. Reduced-order physics-based models were used to calculate thermal, air and mass balance equations and analyze building energy, thermal comfort, pollutant transfer, occupant exposure to pollutants and health impacts, and airborne infection risk. Green building technologies and systems from real and validated practices were collected and standardized as building modules. A Rhino Grasshopper plugin was developed to implement the proposed approach. Coupled simulation between IAQ and energy can be performed and multidiscipline building performance metrics (energy efficiency,

thermal comfort, IAQ, health impacts, and infection risk) were able to be analyzed. Optimization models can be used to find out the optimal design of building modules to achieve certain design goals. The DOE small office prototype, the Syracuse COE case, and the retrofitting case were studied using the developed tool through the proposed approach. Green building modules' performance on energy and IAQ improvement were analyzed. Simulation results suggested that many energy-saving technologies may have impacts on IAQ performance and IAQ strategies can affect energy performance as well. It is essential to comprehensively understand the performance of a certain building module in energy and IAQ performance to achieve the optimal design of a green building. Regarding the office prototype case, the optimal design can save up to 27.8% energy use while mitigating more than 99% infection risk compared to the reference case. Therefore, it reveals that the optimization of green building design using the proposed approach has high potential of energy and IAQ improvement.

A database of building technologies and systems was developed. The performance of IAQ strategies for mitigating infection risk were analyzed. Two emerging technologies were selected and performed chamber experiments and/or CFD simulations to further understand their practical effectiveness, i.e., in-duct needlepoint bipolar ionization (NPBI) units, and the combined use of displacement ventilation and partitions. The in-duct NPBI system can moderately elevate particle removal in the air, with an equivalent SPRE of 3.8-13.6% on particle removal. Ozone generation by the tested in-duct NPBI system can be neglected. No significant VOC removal or generation were observed either. Although its efficiency may sound moderate, its effectiveness on particle removal and infection risk mitigation is high considering its low energy consumption. The combined application of displacement ventilation and desk partitions can effectively mitigate the

potential virus transmission between the infector and the receiver through coughing or talking. However, under the current settings, adding more partitions to form semi-open spaces does not further dramatically increase the ventilation effectiveness. It's likely because of the dominated airflow in the space the thermal plume surrounding the occupants. Particles were less likely to travel across the partitions and reach the receiver's breathing zone from lateral or behind directions. Therefore, for workstations in the office scenario, Partitions that cover the workstation may be sufficient to mitigate the infection transmission between people who are sitting face to face. In the mixing ventilation scenario, because of the cross airflow between cubicles, semi-open space can be more effective in reducing cross-contamination. The abundant performance data from experiments and detailed simulations for the studied technologies will be used by the database of the green building technologies and systems. It will allow these two technologies to be applied through the Green Design Studio approach during the early-design stage for a high-performance building. This can potentially help to address IAQ issues, particularly the airborne transmission of respiratory diseases, while maintaining high energy efficiency.

Future works on this subject include:

- a) Expanding the database of building technologies and introducing more quantitative analyses.
- b) Expanding the use of data-driven building performance simulation models in the platform to further improve the analysis response speed and accuracy.

- c) Applying the proposed tool to more realistic cases and using the measured data to further improve the simulation model with grey-box models.

## 5. Appendix

For the chamber experiments for the needlepoint bipolar ionizers, CO<sub>2</sub> decay tests were performed to determine the infiltration and ventilation rates of the room. CO<sub>2</sub> was injected into the room using the same procedure as the decay tests. Exponential regression was conducted to determine the CO<sub>2</sub> decay rates. Figure A-1 exhibited a decay rate of 0.01h<sup>-1</sup> in the room, indicating a filtration rate of 0.01h<sup>-1</sup>. Figure A-2 showed a decay rate of 1.92h<sup>-1</sup>, which revealed a ventilation rate of 55.6 m<sup>3</sup>/h. Some datapoints in Figure A-2 were missing due to the CO<sub>2</sub> monitor issues.

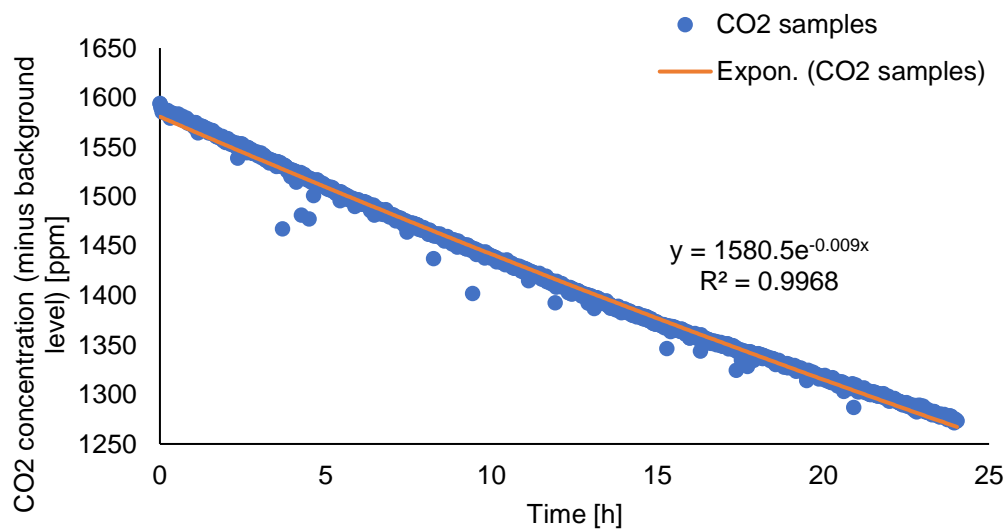


Figure A-1. CO<sub>2</sub> decay test for the infiltration rate (Nov 28<sup>th</sup>, 2021).

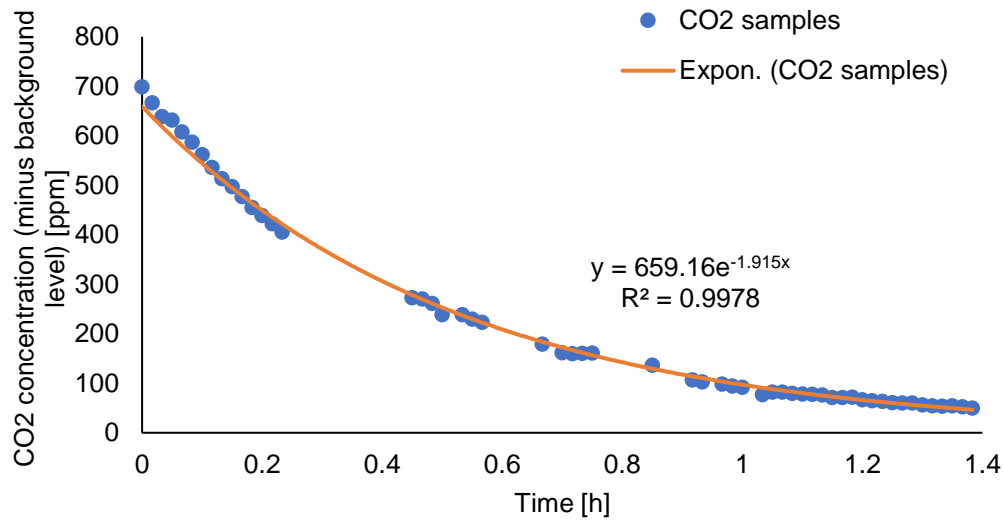


Figure A-2. CO2 decay test for the ventilation rate (Feb 14<sup>th</sup>, 2022).

The ozone decay test was performed by placing an air cleaner that can generate ozone in the test room on May 19<sup>th</sup>, 2021. The experiment was conducted with the same room settings (manikin settings, building materials, furniture etc.). The ventilation rate during the test was estimated through a CO2 test, indicating a ventilation rate of 22.9 m<sup>3</sup>/h (i.e. 0.8 h<sup>-1</sup>). The indoor ozone declined rapidly after the ozone generator (air cleaner) was turned off. The exponential regression of ozone concentration indicated an ozone decay rate of 6.3 h<sup>-1</sup> (Figure A-3). Therefore, the ozone deposition rate on surfaces and by gaseous chemicals was around 5.5 h<sup>-1</sup>.

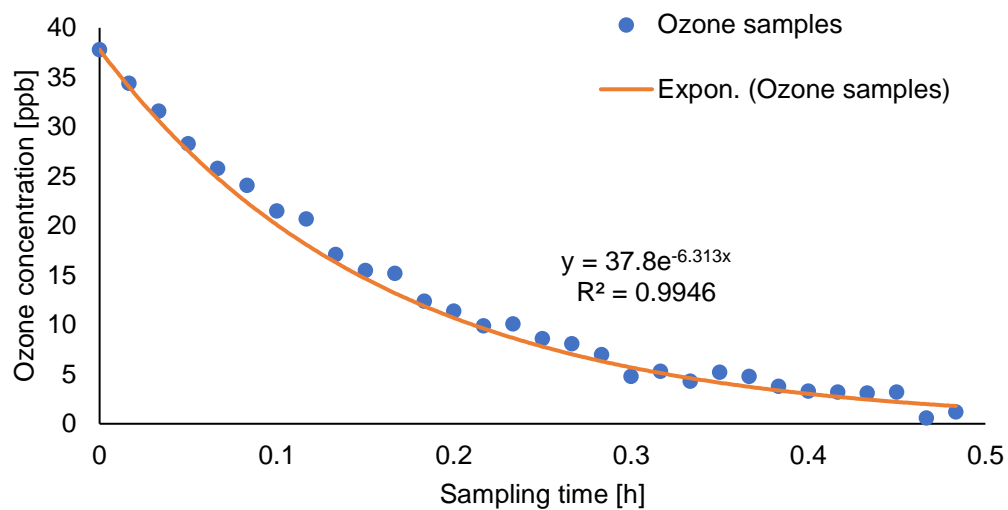


Figure A-3. Ozone decay test (May 19<sup>th</sup>, 2021).

VOC concentrations were measured during the chamber experiments for the ionizers, Table A-1 shows VOC concentrations at different test conditions before and after using the ionizers. Table A-2 shows VOC concentrations of the background level.

Table A-1. VOC concentrations at different test conditions (n.d. = not detected).

Compounds	Formula	VOC concentration at different test conditions [μg/m³]							% change after using ionizer	Relative SD of reference bkg VOC levels <sup>c</sup> [%]
		Bkg	Blower on (with filter)			Ionizer A on				
			Set 1 <sup>b</sup>	Set 2	Mean	Set 1	Set 2	Mean		
Formaldehyde <sup>a</sup>	CH2O	20.6	18.4	19.8	<b>19.1</b>	19.5	22.1	<b>20.8</b>	<b>+8.9%</b>	4.3%
Acetaldehyde	C2H4O	6.6	5.2	5.0	<b>5.1</b>	4.9	4.3	<b>4.6</b>	<b>-9.8%</b>	12.5%
Acetone	C3H6O	9.1	9.0	9.4	<b>9.2</b>	10.1	10.4	<b>10.3</b>	<b>+12.0%</b>	10.5%
Benzene	C6H6	1.8	1.8	1.7	<b>1.8</b>	1.8	1.8	<b>1.8</b>	<b>0.0%</b>	66.7%
Toluene <sup>a</sup>	C7H8	138.7	141.5	139.9	<b>140.7</b>	144.1	150.3	<b>147.2</b>	<b>+4.6%</b>	4.0%
Cyclotrisiloxane, hexamethyl-	C6H18O3Si3	3.8	3.1	3.2	<b>3.2</b>	3.2	3.6	<b>3.4</b>	<b>+6.2%</b>	25.0%
Hexanal	C6H12O	2.5	2.3	2.3	<b>2.3</b>	2.4	2.7	<b>2.6</b>	<b>+13.0%</b>	6.0%

Ethylbenzene	C8H10	n.d.	n.d.	n.d.	n.d.	1.1	1.2	<b>1.2</b>	+	n.d.
Cyclotetrasiloxane, octamethyl-	C8H24O4Si4	2.8	1.6	1.6	<b>1.6</b>	1.6	1.8	<b>1.7</b>	<b>+6.2%</b>	38.8%
Benzaldehyde	C7H6O	2.2	1.8	2.0	<b>1.9</b>	2.3	2.2	<b>2.2</b>	<b>+15.8%</b>	207.7%
2(5H)-Furanone, 3- methyl-	C5H6O2	2.2	1.8	n.d.	<b>1.8</b>	n.d.	n.d.	n.d.	-	n.d.
Phenol	C6H6O	1.0	0.8	0.9	<b>0.9</b>	0.9	0.9	<b>0.9</b>	<b>0.0%</b>	12.5%
Cyclopentasiloxane , decamethyl-	C10H30O5Si5	1.4	1.1	1.4	<b>1.3</b>	1.4	2.1	<b>1.7</b>	<b>+30.8%</b>	18.8%
Acetophenone	C8H8O	2.5	1.5	1.0	<b>1.2</b>	1.0	1.1	<b>1.1</b>	<b>-8.3%</b>	32.0%
Nonanal	C9H18O	1.3	1.1	1.1	<b>1.1</b>	1.5	1.8	<b>1.7</b>	<b>+54.5%</b>	47.8%
Naphthalene	C10H8	2.2	2.0	1.7	<b>1.9</b>	1.5	1.9	<b>1.7</b>	<b>-10.5%</b>	n.d.
Propanoic acid, 2- methyl-, 1-(1,1-	C16H30O4	4.1	5.3	5.6	<b>5.4</b>	5.7	5.4	<b>5.6</b>	<b>+3.7%</b>	27.6%



dimethylethyl)-2-  
methyl-1,3-  
propanediyl ester

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TVOC	/	202.8	198.3	196.6	<b>198.5</b>	203.0	213.6	<b>208.5</b>	<b>+5.0%</b>	/
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<sup>a</sup> Injected compounds.

<sup>b</sup> Two duplicated samples were collected for each sampling set. The average of two duplicated samples was calculated and presented in the table.

<sup>c</sup> From Table A-2.

Table A-2. Reference background concentrations of VOCs concerned in this study.

Compounds	Formula	Reference background concentrations [ $\mu\text{g}/\text{m}^3$ ]						
		Test R1	Test R2	Test R3	Test R4	Mean	SD	Relative
								SD [%]
Formaldehyde <sup>a</sup>	CH <sub>2</sub> O	/	/	/	/	57.8	2.5	4.3
Acetaldehyde	C <sub>2</sub> H <sub>4</sub> O	4.8	6.4	5.9	5.2	5.6	0.7	12.5
Acetone	C <sub>3</sub> H <sub>6</sub> O	4.8	6.0	6.3	5.7	5.7	0.6	10.5
Benzene	C <sub>6</sub> H <sub>6</sub>	2.0	n.d.	1.4	1.2	1.2	0.8	66.7
Toluene <sup>a</sup>	C <sub>7</sub> H <sub>8</sub>	/	/	/	/	107.3	4.3	4.0
Cyclotrisiloxane, hexamethyl-	C <sub>6</sub> H <sub>18</sub> O <sub>3</sub> Si <sub>3</sub>	3.2	6.0	6.1	5.4	5.2	1.3	25.0
Hexanal	C <sub>6</sub> H <sub>12</sub> O	5.4	4.7	5.2	4.7	5.0	0.3	6.0
Ethylbenzene	C <sub>8</sub> H <sub>10</sub>	n.d.	n.d.	n.d.	n.d.	n.d.	n.d.	n.d.
Cyclotetrasiloxane, octamethyl-	C <sub>8</sub> H <sub>24</sub> O <sub>4</sub> Si <sub>4</sub>	2.2	6.3	6.0	5.0	4.9	1.9	38.8

Benzaldehyde	C7H6O	0.53	n.d.	n.d.	n.d.	0.13	0.27	207.7
2(5H)-Furanone, 3-methyl-	C5H6O2	n.d.	n.d.	n.d.	n.d.	n.d.	n.d.	n.d.
Phenol	C6H6O	1.9	1.6	1.5	1.4	1.6	0.2	12.5
Cyclopentasiloxane, decamethyl-	C10H30O5Si5	1.3	1.9	1.8	1.4	1.6	0.3	18.8
Acetophenone	C8H8O	3.5	1.8	2.4	2.1	2.5	0.8	32.0
Nonanal	C9H18O	11.5	3.9	5.6	6.8	6.9	3.3	47.8
Naphthalene	C10H8	n.d.	n.d.	n.d.	n.d.	n.d.	n.d.	n.d.
Propanoic acid, 2-methyl-, 1-(1,1-dimethylethyl)-2-methyl-1,3-propanediyl ester	C16H30O4	3.6	2.4	2.0	3.5	2.9	0.8	27.6

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<sup>a</sup> Injected compounds. The reference background concentrations were sampled through real-time PTR-MS sampling in two injection tests with same settings as Test 10.

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## 7. Index

$A$	Envelope area
$A_{PV}$	Net area of PV surface
$c_i$	Conversion factor of virus quanta emission rate
$c_v$	Viral load in the sputum
$C_0$	Initial pollutant concentration
$C_{adj,j}$	Pollutant concentration of the adjacent room $j$
$C_{es}$	Thermal capacity on exterior envelope node
$C_{exhaust}$	Concentration in the exhaust air
$C_{exposure}$	Exposure-related concentration
$C_{hvac,r}$	Pollutant concentration of the return air of the HVAC system
$C_{hvac,s}$	Pollutant concentration of the supply air of the HVAC system
$C_i$	Indoor concentration of the specific pollutant in room $i$
$C_{in}$	Thermal capacity on indoor air node
$C_{is}$	Thermal capacity on interior envelope node
$C_{out}$	Outdoor concentration of the specific pollutant
$C_{p,air}$	Specific heat of indoor air
$C_{p,w}$	Specific heat of wall material

$C_t$	Indoor pollutant concentration at time $t$
$C_{vir}$	Concentration of viral particles
$d_p$	Particle diameter
$E'$	Emission rate of indoor pollutant source
$E_{clg}$	Energy use by cooling coils in HVAC system
$E_{eqm}$	Energy use by equipment
$E_{fac}$	Energy use by facilities
$E_{fan}$	Energy use by fan in HVAC system
$E_{htg}$	Energy use by heating coils in HVAC system
$E_{lgt}$	Energy use by lighting system
$E_{net}$	Net energy use for building needs
$E_{pro}$	Energy production by renewable or sustainable energy generation system
$E_i$	Total concentration-independent emission rate of the pollutant in room $i$
$ER$	Emission rate of infectious quanta per infector
$f_{activ}$	Fraction of surface area with active solar cells
$f_{cl}$	Clothing area factor and dimensionless
$f_{recirculated}$	Fraction of recirculated air in the supply air of HVAC system
$f_{UV}$	Fraction of UVGI operation time

$F_{air}$	View factor of wall surface to air temperature
$F_{gnd}$	View factor of wall surface to ground surface temperature
$F_{sky}$	View factor of wall surface to sky temperature
$G_T$	Total solar radiation incident on PV array
$h$	Heat transfer coefficient
$h_c$	Overall convective heat transfer coefficient between body (including clothing) and the surrounding air
$h_{es}$	Convective heat transfer coefficient on the exterior surface
$h_{is}$	Convective heat transfer coefficient on the interior surface
$h_{r,gnd}$	Equivalent radiative heat transfer coefficient between envelope surface and ground
$h_{r,sky}$	Equivalent radiative heat transfer coefficient between envelope surface and sky
$h_{r,air}$	Equivalent radiative heat transfer coefficient between envelope surface and air
$H$	Room height
$I$	Number of infectors
$IR$	Inhalation rate
$I_{cl}$	Clothing vapor permeation efficiency ratio of actual evaporative heat flow capability through clothing to sensible heat flow capability
$IR_H$	Horizontal infrared radiation intensity
$k$	Thermal conductivity

$k_{AC}$	Pollutant removal rate by air cleaning units
$k_{AirCleaner}$	Fresh air supply rate by standalone portable air cleaners
$k_{a,i}$	First-order reaction rate of the pollutant with other gaseous compounds in room $i$
$k_{b,i}$	First-order generation rate of the pollutant due to reactions in room $i$
$k_{dep}$	Deposition/removal rate of the pollutant in the room
$k_{deposition}$	Infectious particle deposition rate
$k_{d,i}$	Deposition rate of the pollutant in room $i$
$k_{filter}$	Pollutant removal by filtration
$k_{inactivation}$	Pathogen natural inactivation rate in the air
$k_{ionizer}$	Pollutant removal by ionization
$k_n$	Natural decay rate of pollutant
$k_{UV}$	Pathogen inactivation rate by UVGI system
$K_{filter}$	Filter pressure drop coefficient
$L$	Thickness of envelope
$m'_{adj,j}$	Air flow rate between room $i$ and the adjacent room $j$
$m'_{hvac,i}$	Supply and return air flow rate of the HVAC system to room $i$
$m'_{inf}$	Air infiltration rate
$m'_{inf,i}$	Air flow rate of air infiltration or natural ventilation between the outdoor air and the indoor air of room $i$

$m'_{PAC,i}$	Air flow rate of portable air cleaners in room $i$
$N_C$	New infection case number
$N_D$	Particle number
$N_S$	Susceptible case number
$p_a$	Water vapor pressure of indoor air
$P$	Infection possibility
$P_i$	Penetration factor of the pollutant through infiltration or natural ventilation
$P_{PV}$	Electrical power produced by PV
$q'_{DHI,ext}$	Heat flow due to incident DHI on the exterior side of envelope
$q'_{DNI,ext}$	Heat flow due to incident DNI on the exterior side of envelope
$q'_{HL,1}$	Heat loss through skin
$q'_{HL,2}$	Heat loss through sweating
$q'_{HL,3}$	Latent respiratory heat loss
$q'_{HL,4}$	Dry respiratory heat loss
$q'_{HL,5}$	Heat loss by radiation
$q'_{HL,6}$	Heat loss by convection
$q'_{infiltration}$	Heat flow from air infiltration on indoor air
$q'_{internal}$	Internal heat gain/loss of indoor air



$q'_{LWR,ext}$	Longwave radiative heat flow on exterior envelope
$q'_{LWR,s-air}$	Longwave radiative heat flow on exterior envelope due to radiation exchange with outdoor air
$q'_{LWR,s-gnd}$	Longwave radiative heat flow on exterior envelope due to radiation exchange with ground
$q'_{LWR,s-sky}$	Longwave radiative heat flow on exterior envelope due to radiation exchange with sky
$q'_{MW}$	Heat generation by individuals
$q'_{rad,in}$	Radiative heat flow on interior surface of envelope
$q'_{rad,out}$	Radiative heat flow on exterior surface of envelope
$q'_{sol-reflected,ext}$	Heat flow due to ground-reflected solar radiation on the exterior side of envelope
$q'_{sol,in}$	Solar heat flow on interior surface of envelope
$q'_{sol,out}$	Solar heat flow on exterior surface of envelope
$q'_{system}$	Heat flow from HVAC system on indoor air
$q'_{WME}$	Heat flow from external work
$Q$	Air flow rate
$Q_{AC}$	Air flow rate of air cleaner
$Q_{inhale}$	Inhalation rate
$Q_{intake}$	Volume rate of air intake

$R_{es}$	Resistance between external surface and exterior air
$R_i$	Total concentration-independent removal rate of the pollutant in room $i$
$R_{is}$	Resistance between internal surface and interior air
$R_{is,adj,l}$	Resistance between internal surface of the adjacent zone and interior air
$R_w$	Resistance of envelope
$t$	Time
$t_s$	Sampling time
$ts$	Thermal sensation transfer coefficient
$T_{cl}$	Surface temperature of clothes
$T_{es}$	Surface temperature of envelope exterior side
$T_{gnd}$	Ground surface temperature
$T_{hvac,s}$	Supply air temperature of HVAC system
$T_{in}$	Indoor air temperature
$T_{is}$	Surface temperature of envelope interior side
$T_{is,adj,l}$	Surface temperature of envelope interior side of the adjacent zone
$T_{mrt}$	Mean radiant temperature
$T_{out}$	Outdoor air temperature
$T_{sky}$	Sky temperature

$u$	Input vector of SSM (input variables)
$v$	Control vector of SSM (control variables)
$V$	Room volume
$V_D$	Particle volume
$V_i$	Volume of room $i$
$V'_{CO2}$	Volumetric emission rate of CO2
$w$	Input vector of SSM (input variables)
$x$	State vector of SSM (state variables)
$v_{met}$	Wind speed measured at the meteorological station
$v_z$	Wind speed at altitude $z$
$y$	Output vector of SSM (output variables)
$y_0$	Baseline prevalence of illness per year
$z$	Altitude or height above ground
$z_{met}$	Height above ground of the wind speed sensor at the meteorological station
$\alpha$	Absorptance of envelope
$\beta$	Factor to split the view factor between surface and sky
$\beta_{exp}$	Coefficient of concentration change associated with exposure
$\alpha_{wind}$	Wind speed profile exponent at the site

$\alpha_{met}$	Wind speed profile exponent at the meteorological station
$\delta$	Wind speed profile boundary layer thickness at the site
$\delta_{met}$	Wind speed profile boundary layer thickness at the meteorological station
$\Delta P_{coil}$	Pressure drops by heating/cooling coils in HVAC system
$\Delta P_{diffuser}$	Pressure drops by air diffuser in HVAC system
$\Delta P_{filter}$	Pressure drops by filter in HVAC system
$\Delta P_{hvac}$	Total pressure drops in HVAC system
$\Delta P_{return duct}$	Pressure drops by return ducts in HVAC system
$\Delta P_{return grille}$	Pressure drops by return grille in HVAC system
$\Delta P_{supply duct}$	Pressure drops by supply ducts in HVAC system
$\Delta t$	Exposure time
$\varepsilon$	Longwave emittance of surface
$\varepsilon_{hvac,rec}$	recirculated air ratio of HVAC system
$\varepsilon_{vent}$	Ventilation effectiveness
$\eta_{AirCleaner}$	Particle removal efficiency by air cleaner
$\eta_{cell}$	Module conversion efficiency of PV
$\eta_{coil}$	Conversion factor of HVAC coils
$\eta_{filter}$	Particles filtration efficiency by filter

$\eta_{invert}$	DC to AC conversion efficiency of PV
$\eta_{PAC,i}$	Pollutant removal efficiency of portable air cleaners in room $i$
$\eta'_{mask}$	Equivalent fraction of infectious particle penetration through the face mask
$\theta_{in}$	Angle of incidence
$\theta_{s-sky}$	Angle between surface and sky
$\theta_z$	Sun's zenith angle
$\lambda_{HVAC}$	Fresh air supply rate by the HVAC system
$\lambda_{inf}$	Air infiltration rate
$\lambda_Q$	Ventilation rate by the control system
$\lambda_{recirculated}$	Recirculated airflow rate
$\lambda_{vent}$	Equivalent ventilation air change rate
$A$	Equivalent room air change rate
$\rho$	Reflectance of envelope
$\rho_{air}$	Air density
$\rho_{gnd}$	Ground reflectance
$\rho_w$	Density of wall material
$\sigma$	Stefan-Boltzmann constant
$\tau$	Transmittance of envelope

$\Phi$	Tilt angle of the surface
ACD	Air cleaning device
ACH	Air change per hour
ADAF	Age-dependent adjustment factor
AHU	Air handling unit
ANN	Artificial neural network
API	Application programming interface
AQI	Air quality index
BCVTB	Building Controls Virtual Test Bed
BMR	Basal metabolic rate
BPO	Building performance optimization
BPS	Building performance simulation
BREP	Boundary representation
CADR	Clean air delivery rate
CFD	Computational fluid dynamics
CO	Carbon monoxide
CO <sub>2</sub>	Carbon dioxide
COP	Coefficient of performance

C-R	Concentration-response
CV	Coefficient of variation
DALYs	Disability-adjusted life years
DCV	Demand-controlled ventilation
DHI	Diffuse horizontal irradiation
DHW	Domestic hot water
DPM	Discrete phase model
DNI	Direct normal irradiation
DNPH	2,4-Dinitrophenylhydrazine
EPS	Expanded polystyrene
EPW	Energyplus weather file
ERV	Energy recovery ventilator
EUI	Energy use intensity
FMI	Functional Mock-up Interface
FMU	Functional Mock-up Unit
GA	Genetic algorithm
GC	Gas chromatograph
GDS	Green Design Studio

HCHO	Formaldehyde
HEPA	High efficiency particulate air (filter)
HPB	High-performance building
HPLC	High-performance liquid chromatography
HVAC	Heating, ventilation, and air conditioning
IAQ	Indoor air quality
ID	Intake-DALY
IDF	Input data file of EnergyPlus
IEQ	Indoor environmental quality
IND	Intake-incidence-DALY
IOT	Internet of thing
M	Metabolic rate
MERV	Minimum efficiency reporting value
ML	Machine learning
MLR	Multiple linear regression
MOGA	Multi-objective genetic algorithm
MOO	Multi-objective optimization
MPC	Model predictive control



MS	Mass selective detector
NO <sub>2</sub>	Nitrogen dioxide
NO <sub>x</sub>	Nitrogen oxides
NPBI	Needlepoint bipolar ionization
O <sub>3</sub>	Ozone
OLS	Ordinary least squares regression
OSM	OpenStudio model file
PAC	Portable air cleaner
PCM	Phase-change material
PCO	Photocatalytic oxidation
PM	Particulate matter
PM <sub>2.5</sub>	Particulate matters with a diameter of 2.5 micrometers or less
PM <sub>10</sub>	Particulate matters with a diameter of 10 micrometers or less
PMV	Predicted mean vote
ppb	Part per billion
ppm	Part per million
PPD	Predicted percentage of dissatisfied
PPE	Personal protective equipment

PTR-MS	Proton transfer reaction - Mass spectrometry
PV	Photovoltaics
RC	Resistance-capacitance
RH	Relative humidity
SET	Standard effective temperature
SHGC	Solar heat gain coefficient
SIMPLE	Semi-implicit method for pressure-linked equations
SO <sub>2</sub>	Sulfur dioxide
SPRE	Single-pass pollutant removal efficiency
SSM	State-space matrices/models
SVM	Support vector machine
SVOC	Semi-volatile organic compound
TVOC	Total volatile organic compound
UV	Ultraviolet
UVGI	Ultraviolet germicidal irradiation
VAV	Variable air volume
VIP	Vacuum insulation panel
VOC	Volatile organic compound

YLD       Years of life disability

YLL       Years of life lost

## 8. Vita

# Jialei Shen

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### Research Objectives & Interests

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Indoor air quality (IAQ) and ventilation  
Building energy and environmental performance simulation and optimization  
Urban morphology, greening, transportation, and urban microclimate  
Evaluation and mitigation of infection risk through airborne transmission

### Education

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2018 – 2023	Syracuse University	Ph.D.	Mechanical & Aerospace Engineering
2015 – 2018	Nanjing University	M.S.	Architectural Engineering
2011 – 2015	Xi'an Jiaotong University	B.S.	Mechanical Engineering

### Experience

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<u>After graduation</u>			
2022 – 2023	NIST & University of Maryland	Research Associate	Infection risk estimation in CONTAM; ANT: CONTAM on Rhino Grasshopper (funded by NIST)
<u>Before graduation</u>			
2022 Summer	Syracuse University	RA	IAQ Sensor testing and evaluation (Industrial project)
2021 – 2022	Syracuse University	RA	Effectiveness of Building Systems Strategies for Mitigation of Airborne Transmission of SARS-CoV-2 (Industrial project)
2020 Summer	Syracuse University	RA	Integrated Whole-Building Energy Efficiency Retrofit Solution for Residences in Cold/Very Cold Climates (DOE ABC project)
2020 Spring	Syracuse University	TA	ECS104: Engineering Computational Tools
2019 Fall	Syracuse University	TA	MEE416: Mechanical Engineering Lab

2019 Summer	Syracuse University	RA	Green building design for high-performance buildings (funded by Syracuse Center of Excellence)
2017 – 2018	Nanjing University	RA	Research on Residential Building Ventilation and IAQ Strategies (NSFC project)
2016 – 2017	Nanjing University	RA	Urban Form-Microclimate Coupling Mechanism and Control (NSFC project)

**Publications** (total citations: 550, based on Google Scholar)

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### Book chapters

#### 2022

1. Zhang, J., **Shen, J.**, Gao, Z., 2022. Managing IAQ at Multiple Scales - from Urban to Personal Microenvironments. *Handbook of Indoor Air Quality*, 1-42.

### Journal articles

#### 2023

16. Xu, F., **Shen, J.**, Gao, Z., 2023. A field measurement study of the effects of outdoor pollutants and room volumes on indoor fine particle and ozone concentrations in residential buildings in Nanjing. *Journal of Building Engineering*. (co-first author)

#### 2022

15. Guo, X., Buccolieri, R., Gao, Z., Zhang, M., Lyu, T., Rui, L., **Shen, J.**, 2022. On the effects of urban-like intersections on ventilation and pollutant dispersion. *Building Simulation* 15 (3), 419-433.

#### 2021

14. **Shen, J.**, Kong, M., Dong, B., Birnkrant, M.J., Zhang, J., 2021. Airborne transmission of SARS-CoV-2 in indoor environments: A comprehensive review. *Science and Technology for the Built Environment* 27 (10), 1331-1367.
13. Guo, X., Gao, Z., Buccolieri, R., Zhang, M., **Shen, J.**, 2021. Effect of greening on pollutant dispersion and ventilation at urban street intersections. *Building and Environment* 203, 108075.
12. **Shen, J.**, Kong, M., Dong, B., Birnkrant, M.J., Zhang, J., 2021. A systematic approach to estimating the effectiveness of multi-scale IAQ strategies for reducing the risk of airborne infection of SARS-CoV-2. *Building and Environment* 200, 107926.
11. Peng, Y., Gao, Z., Buccolieri, R., **Shen, J.**, Ding, W., 2021. Urban ventilation of typical residential streets and impact of building form variation. *Sustainable Cities and Society* 67, 102735.
10. Zhang, M., Gao, Z., Guo, X., **Shen, J.**, 2021. Ventilation and pollutant concentration for the pedestrian zone, the near-wall zone, and the canopy layer at urban intersections. *International Journal of Environmental Research and Public Health* 18 (21), 11080.

#### 2020

9. **Shen, J.**, Krietemeyer, B., Bartosh, A., Gao, Z., Zhang, J., 2020. Green Design Studio: A modular-based approach for high-performance building design. *Building Simulation* 14, 241–268.

#### 2019

8. Guo, C., Gao, Z., **Shen, J.**, 2019. Emission Rates of Indoor Ozone Emission Devices: A Literature Review. *Building and Environment* 158, 302–318.
7. **Shen, J.**, Gao, Z., 2019. Commuter Exposure to Particulate Matters in Four Common Transportation Modes in Nanjing. *Building and Environment* 156, 156–170.
6. Huang, Y., Rui, L., Gao, Z., **Shen, J.**, 2019. Research progress on reaction between ozone and human occupants. *Journal of Environment and Health* 36(02), 182–187. (in Chinese language)

#### 2018

5. You, W., **Shen, J.**, Ding, W., 2018. Improving residential building arrangement design by assessing outdoor ventilation efficiency in different regional spaces. *Architectural Science Review*.
4. Yang, Z., **Shen, J.**, Gao, Z., 2018. Ventilation and Air Quality in Student Dormitory in China: A Case Study during Summer in Nanjing. *International Journal of Environmental Research and Public Health* 15(7), 1328.
3. Rui, L., Buccolieri, R., Gao, Z., Ding, W., **Shen, J.**, 2018. The Impact of Green Space Layouts on Microclimate and Air Quality in Residential Districts of Nanjing, China. *Forests* 9, 224.
2. **Shen, J.**, Gao, Z., 2018. Ozone removal on building material surface: A literature review. *Building and Environment* 134, 205–217.

#### 2017

1. **Shen, J.**, Gao, Z., Ding, W., Yu, Y., 2017. An investigation on the effect of street morphology to ambient air quality using six real-world cases. *Atmospheric Environment* 164, 85–101.

### **Conference articles and presentations**

#### 2023

17. Hu, Y., **Shen, J.**, Zhang, J., Gao, Z., 2023. A CFD approach to reduce the risk of Covid-19 airborne transmission in a typical office. *IAQVEC 2023*, Tokyo, Japan.

#### 2022

16. **Shen, J.**, Nabutola, K., Birnkrant, M.J., McKinney, P.J., Dong, B., Zhang, J., 2022. Estimation of infection risk through airborne transmission in large open spaces with different air distributions. *RoomVent 2022*, Xi'an, China. (Best Paper Award)
15. Liu, Z., **Shen, J.**, Guo, B., Zhang, J., 2022. Modeling Secondary Emissions from Indoor Chemical Reactions Initiated by Ozone and Hydroxyl Radicals. *Indoor Air 2022*, Kuopio, Finland.
14. Liu, Z., Parry-Nweye, E., Dhaouadi, Y., Wang, X., **Shen, J.**, Guo, B., Ramirez, M., Gupta, B., Ren, D., Dong, B., Zhang, J., 2022. A Comprehensive Approach to Testing Air Cleaning Devices Under Realistic Room Conditions. *Indoor Air 2022*, Kuopio, Finland.
13. **Shen, J.**, Liu, Z., Guo, B., Love, D., Dekold, M., Birnkrant, M.J., Dong, B., Zhang, J., 2022. Performance of in-duct bipolar ionization devices on pollutant removal and potential byproduct formation in indoor environments. *Indoor Air 2022*, Kuopio, Finland.

#### 2021

12. **Shen, J.**, Liu, Z., Hu, Y., Birnkrant, M.J., Gao, Z., Dong, B., Zhang, J., 2021. Effectiveness of combined displacement ventilation and partitions for reducing airborne transmission of respiratory diseases in office settings. *Healthy Buildings 2021 America*, Honolulu, United States (online).
11. **Shen, J.**, Kong, M., Dong, B., Birnkrant, M.J., Zhang, J., 2021. Effectiveness of IAQ strategies for mitigating the airborne infection risk of SARS-CoV-2. *IBPC 2021*, Copenhagen, Denmark (online).

10. **Shen, J.**, 2021. Ozone Emission Sources and Surface Removal in Indoor Environments. *ASHRAE Virtual Annual Conference 2021 - Seminar 14: Reducing Ozone: A Critical Factor in Improving IAQ* (online).

#### 2019

9. **Shen, J.**, Zhang, J., 2019. An approach to develop a green technology database for residential buildings. *IAQVEC 2019*, Bari, Italy.
8. Zhang, J., **Shen, J.**, Krietemeyer, E., Bartosh, A., Gu, R., 2019. Green Design Studio for Healthy and Sustainable Buildings and Communities: Exploration of an Innovative Approach. *17th New York State Green Building Conference*, Syracuse, United States.
7. **Shen, J.**, Gu, R., Bartosh, A., Krietemeyer, E., Zhang, J., 2019. Modular-based Green Design Studio for Sustainable Building. *17th New York State Green Building Conference*, Syracuse, United States.

#### 2017

6. **Shen, J.**, Chen, J., Zhang, X., Zou, S., Gao, Z., 2017. Outdoor and Indoor Ozone Concentration Estimation Based on Artificial Neural Network and Single Zone Mass Balance Model. *10<sup>th</sup> International Symposium on Heating, Ventilation and Air Conditioning (ISHVAC 2017)*, Jinan, China.
5. **Shen, J.**, Gao, Z., 2017. Commuter Exposure to Particulate Matter of Different Transportation Modes in Nanjing. *10<sup>th</sup> International Symposium on Heating, Ventilation and Air Conditioning (ISHVAC 2017)*, Jinan, China.
4. **Shen, J.**, Gao, Z., 2017. Steady State and Dynamic Indoor Ozone Estimation Model with Indoor Ozone Source. *Healthy Buildings 2017 Europe*, Lublin, Poland.
3. **Shen, J.**, Gao, Z., Ding, W., You, W., 2017. Study on Relationship of Residential Building Form Variations to Pollutant Dispersion in Streets. *Healthy Buildings 2017 Europe*, Lublin, Poland.
2. You, W., **Shen, J.**, Ding, W., 2017. Improving Wind Environment of Residential Neighborhoods by Understanding the Relationship between Building Layouts and Ventilation Efficiency. *8<sup>th</sup> International Conference on Applied Energy (ICAE 2016)*, Beijing, China.
1. You, W., **Shen, J.**, Ding, W., 2017. Improving wind environment design based on assessing spatial distribution of ventilation efficiency in regional space. *9<sup>th</sup> International Conference on Applied Energy (ICAE 2017)*, Cardiff, United Kingdom.

#### **Grants & Fellowships**

2018, 2020	Graduate Fellowship	Syracuse University
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#### **Honors & Awards**

2022	Best Paper Award	RoomVent 2022 Conference
2019	Excellent Master's Degree Dissertation of Jiangsu Province	Ministry of Education of Jiangsu Province, China
2018	Excellent Postgraduate Honor	Nanjing University
2017	National Scholarship	Ministry of Education of China

2017	ISIAQ Student Conference Support Award	International Society of Indoor Air Quality and Climate (ISIAQ)
2017	Excellent Graduate Student Honor	Nanjing University
2014	Excellent Social Activist Honor	Xi'an Jiaotong University
2013	Excellent Undergraduate Student Honor	Xi'an Jiaotong University
2012 – 2014	Siyuan Scholarship	Xi'an Jiaotong University

### Services

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2020 – Now	Serving as a reviewer for journals including 1) <i>Building and Environment</i> , 2) <i>Building Simulation</i> , 3) <i>Science and Technology for the Built Environment</i> , 4) <i>International Journal of Ventilation</i> , etc.
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### Memberships & Affiliations

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2020 – Now	<i>Science and Technology for the Built Environment</i> journal	Reviewer Board
2018 – Now	American Society of Heating, Refrigerating and Air-Conditioning Engineers (ASHRAE)	Associate Member
2022 – Now	International Building Performance Simulation Association (IBPSA)	Associate Member
2019 – 2022	ASHRAE Syracuse University Student Branch	Vice President
2021 – 2023	International Society of Indoor Air Quality and Climate (ISIAQ)	Student Member