

On the use of the Material Point Method for large rotation problems

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www.screwpilesforoffshorewind.co.uk

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(implicit) Material Point Method focus at Durham University

seabed ploughing (Cortis) screwpile installation (Wang)

overcoming volumetric locking (CMAME, 2018) IGA-based MPM (Ghaffari-Motlagh) B-spline representation & enforcement of boundaries (Bing) generalised interpolation & gradient plasticity (Charlton)

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- Designing foundations for offshore wind turbines is challenging because of the complex dynamic mechanical loading environment;
- monopiles are currently the most commonly used foundation in the offshore wind market due to their ease of installation;
- this research is part of a larger UK research council funded grant investigating alternative foundation solutions for offshore wind.





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- The research aims to make screw piles a more attractive foundation (or anchoring) option for offshore wind farms;
- installation torque in different seabed conditions is a key question;
- computational modelling of screw pile installation is Durham's focus;
- challenging problem: truly 3D, large deformation, non-linear material behaviour -MPM appears to be ideal?





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Implicit material point formulation governing equations



governing equation of elasticity

$$\nabla \sigma_{ij} + f_i{}^b = 0 \quad \text{in} \quad \Omega$$

subject to the following

 $u_i = g_i$ on $\partial \Omega_D$ and $\sigma_{ij} n_j = t_i$ on $\partial \Omega_N$

where g_i and t_i are the Dirichlet and Neumann boundary conditions

discretised into the conventional updated Lagrangian form

$$\int_{\varphi_t(E)} [\nabla S_{vp}]^T \{\sigma\} \mathrm{d}v - \int_{\varphi_t(E)} [S_{vp}]^T \{b\} \mathrm{d}v - \int_{\varphi_t(\partial \Omega_N)} [S_{vp}]^T \{t\} \mathrm{d}s = \{0\}$$

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finite deformation mechanics

linear isotropic relationship is assumed between elastic logarithmic strains and Kirchhoff stresses

$$au_{ij} = D_{ijkl}^{\mathsf{e}} \varepsilon_{kl}^{\mathsf{e}} \qquad \text{where} \qquad \varepsilon_{ij}^{\mathsf{e}} = \frac{1}{2} \ln \left(F_{ik}^{\mathsf{e}} F_{jk}^{\mathsf{e}} \right)$$

and the deformation gradient is obtained as

$$F_{ij} = \frac{\partial x_i}{\partial X_j}$$
 and $F_{ij} = F_{ik}^{\mathsf{e}} F_{kj}^{\mathsf{p}}$

the Cauchy stress is recovered using

$$\sigma_{ij} = \frac{1}{J} \tau_{ij}$$
 where $J = \det(F_{ij})$

the adopted stress and strain measures provide the most straightforward way of implementing large strain elasto-plasticity

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finite deformation mechanics: deformation gradient update

A point of departure of implicit MP methods from conventional finite elements is the calculation of the deformation gradient

$$F_{ij} = \Delta F_{ik} F_{kj}^n$$
 where $\Delta F_{ij} = \delta_{ij} + \frac{\partial \Delta u_i}{\partial \tilde{X}_j}$

and $\tilde{X}_i = x_i - \Delta u_i$ are the coordinates at the start of the loadstep.

However, equilibrium is satisfied in the updated frame, requiring mapping of the shape function derivatives

$$\frac{\partial S_{vp}}{\partial x_i} = \frac{\partial S_{vp}}{\partial \tilde{X}_j} \frac{\partial \tilde{X}_j}{\partial x_i} = \frac{\partial S_{vp}}{\partial \tilde{X}_j} (\Delta F_{ji})^{-1}$$

Note that the spatial derivatives are needed to integrate the stiffness and internal force contribution of a material point in an updated Lagrangian formulation.

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basis functions



$$\begin{split} S_{vp} &= 1 + (\tilde{X}_p - \tilde{X}_v)/h & -h < \tilde{X}_p - \tilde{X}_v \le 0 \\ S_{vp} &= 1 - (\tilde{X}_p - \tilde{X}_v)/h & 0 < \tilde{X}_p - \tilde{X}_v \le h, \end{split}$$

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MPM simulations for large rotation



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Numerics & implicit implementation non-linear solution



fully implicit Newton process used to solve the non-linear equation

$$\{f^{oobf}\} = \{f^{int}\} + \{f^{ext}\} = \{0\}$$

where

$$\{f^{int}\} = \bigwedge_{\forall p} \left(\left[\nabla S_{vp} \right]^T \{\sigma_p\} V_p \right) \text{ and}$$
$$\{f^{ext}\} = \int_{\varphi_t(\partial \Omega_N)} [S_{vp}]^T \{t\} ds + \bigwedge_{\forall p} \left([S_{vp}]^T \{f^b\} V_p \right)$$

global consistent tangent determined analytically for optimal convergence (linearisation of the internal force with respect to the unknown displacements)

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Numerics & implicit implementation computational procedure

For each loadstep:

- 1. assemble the internal force stiffness contribution of all material points;
- increment the external tractions and/or body forces in and solve for the nodal displacements within a loadstep using the Newton process;
- update material point positions, stresses, volumes, domains, etc.;

4. reset or replace the background grid.



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simple stretch (validation)





- $l_0 = 2, h = 1$
- $E = 10^3$, $\nu = 0$
- von Mises, $\rho_y = 400$
- ▶ 2² MPs/element
- plane strain
- moving mesh, edge displacement u/l₀ = 2

simple stretch (validation)





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MPM simulations for large rotation

Numerical examples simple stretch (validation)









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MPM simulations for large rotation

Numerical examples simple stretch (validation)









- $l_0 = 2, h = 1$
- $E = 10^3$, $\nu = 0.4$
- elastic behaviour
- ▶ 2² & 8² MPs/e
- plane strain
- ► moving mesh, corner displacement ∆x, ∆y = 4





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corner stretch







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500

corner stretch

Numerical examples

 2^2 material points A--<u>A</u>--<u>A</u> 400 Nodal reaction force 300 200 100 --A-- CPDI2t 0¢ 0 2 3 4 Displacement



$$l_0 = 2, h = 1$$

•
$$E = 10^3$$
, $\nu = 0.4$

- elastic behaviour
- $2^2 \& 8^2 \text{ MPs/e}$
- plane strain
- moving mesh, corner displacement $\Delta x, \Delta y = 4$













corner stretch







corner stretch

$$\nabla S_{vp} = \frac{1}{2V_p} \left(S_v(\{x_1\}) \left\{ \begin{array}{c} y_2 - y_3 \\ x_3 - x_2 \end{array} \right\} \right. \\ \left. + S_v(\{x_2\}) \left\{ \begin{array}{c} y_3 - y_1 \\ x_1 - x_3 \end{array} \right\} \\ \left. + S_v(\{x_3\}) \left\{ \begin{array}{c} y_1 - y_2 \\ x_2 - x_1 \end{array} \right\} \right) \right] \\ \left. \\ \text{node of interest} \\ \left\{ x_3 \right\} \\ \left\{ \begin{array}{c} x_3 \\ x_1 \\ x_1 \\ x_1 \\ x_1 \\ x_1 \\ x_2 \\ x_1 \\ x_2 \\ x_2$$





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corner stretch

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corner stretch







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corner stretch





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MPM simulations for large rotation

CPDI2t

doughnut twist

rotational moving mesh

doughnut twist

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rotational moving mesh

doughnut twist

- $R_o = 10, R_i = 5$
- $E = 10^6$, $\nu = 0$
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fixed outer boundary and incremental rotation $\Delta \alpha$ on inner boundary with rotational moving mesh

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MPM simulations for large rotation

13 / 15

inner boundary with rotational moving mesh

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Observations

- A unified implicit computational framework for sMPM and CPDIs has been developed;
- moving mesh concept extended to include rotational deformation;
- CPDI approaches reduce the instabilities inherent in material point methods; but
- only the sMPM and CPDI1 approaches obtain physically meaningful solutions for large rotational problems;
- CPDI2q faces issues due to distortion of particle domains; and
- CPDI2t has degenerative cases with spurious spatial derivatives of the basis functions.

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Acknowledgements

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