

MASTER

A design rule to include the effect of leaning columns in steel frame stability

Broeks, Wim M.

Award date:
2021

[Link to publication](#)

Disclaimer

This document contains a student thesis (bachelor's or master's), as authored by a student at Eindhoven University of Technology. Student theses are made available in the TU/e repository upon obtaining the required degree. The grade received is not published on the document as presented in the repository. The required complexity or quality of research of student theses may vary by program, and the required minimum study period may vary in duration.

General rights

Copyright and moral rights for the publications made accessible in the public portal are retained by the authors and/or other copyright owners and it is a condition of accessing publications that users recognise and abide by the legal requirements associated with these rights.

- Users may download and print one copy of any publication from the public portal for the purpose of private study or research.
- You may not further distribute the material or use it for any profit-making activity or commercial gain

A DESIGN RULE TO INCLUDE THE EFFECT OF LEANING COLUMNS IN STEEL FRAME STABILITY

GRADUATION REPORT

29-1-2021

TU/e Eindhoven

Broeks, W.M.

E-mail:

w.m.broeks@student.tue.nl

wm.broeks@hotmail.com

Identity number:

1262947

Graduation Committee:

H.H. Snijder (TU/e)

Graduation Committee:

P. Teeuwen (Witteveen & Bos)

Graduation Committee:

D. Leonetti (TU/e)

Graduation Committee:

H. Hofmeyer (TU/e)



ACKNOWLEDGMENTS

This graduation report has been written to graduate for the degree of Master of Science at Eindhoven University of Technology (TU/e), program Architecture, Building, and Planning (ABP), Structural Design.

I would like to thank the members of my graduation committee, prof. ir. H.H. (Bert) Snijder, dr. ir. P.A. (Paul) Teeuwen, dr. ir. D. (Davide) Leonetti, and dr. ir. H. (Herm) Hofmeyer for their extensive supervision of this research, and the amount of time spent to accomplish this. The knowledge shared in the graduation meetings helped to finish this graduation project. Furthermore, I would like to thank prof. ir. H.H. (Bert) Snijder again for initiating this research.

Lastly, I would like to thank my family, friends, and girlfriend for their moral support.

W. (Wim) Broeks

Eindhoven, January 2021

SUMMARY

The stability of a steel structure is often ensured by one part of a frame, the so-called stabilizing frame. When the steel structure deforms in the lateral direction and the other parts of the steel structure are hinged connected to the stabilizing frame, the vertical loads on the columns that are not part of the stabilizing frame introduce additional lateral loads on the stabilizing frame due to leaning. This phenomenon is called the leaning column effect, or in Dutch 'aanpendelende belasting'. To ensure the resistance of the stabilizing frame, these additional leaning column loads have to be taken into account by the design rules. Back in the days, the NEN series were in force. In the code NEN 6771, the effect of the leaning column is included in the load side of the design rule by an equivalent total force F_{tot} . However, the background for this design rule is unknown. Nowadays, the Eurocode (EC) series are in force and ensure the structural safety of all structures. In the EC EN 1993-1-1, it appears that this leaning column effect is not included in the design rule.

The aim of this graduation project is to numerically verify the design rule of the NEN 6771, where they included the leaning column effect in the load side of the design rule, and rewrite the design rule of the EN 1993-1-1 to also include the leaning column effect. First, the design rule of the EN 1993-1-1 is rewritten with the same content as the NEN 6771, but the same format as the current EN 1993-1-1 design rule. After that, the ultimate resistance loads obtained with the various design rules will be compared with the ultimate loads determined with various analyses with the Finite Element Method (FEM). As a starting point of the research, the most simplistic form of a stabilizing frame with a leaning column was taken, as illustrated in Figure 1.

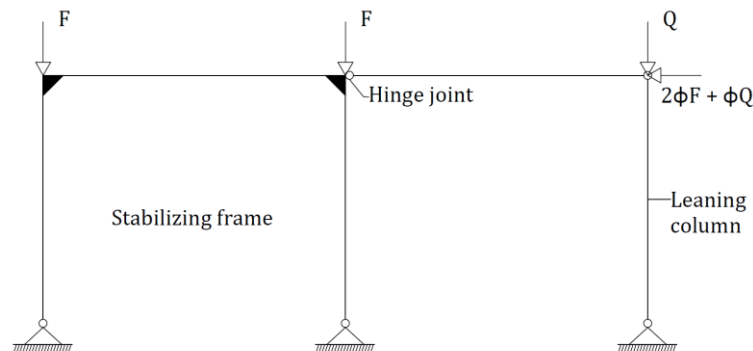


Figure 1: Frame configuration with a stabilizing frame and a leaning column

The Finite Element (FE) Model has to reflect reality in the best way. To achieve this, a study has been performed to create the most realistic model and in the end, this model is translated into an actual model including the leaning column effect. The analysis that should determine the ultimate load that the frame can carry also should reflect reality in the best way. Therefore various analysis methods were studied and performed by the FE-Model. To validate the accuracy of the way of modeling and the analysis methods, a validation study was performed for a sway frame without a leaning column.

After validating the model, the various analyses are performed again for a frame including the leaning column effect and this is processed in a case study. The numerical results of the various analyses are compared to the ultimate resistance load according to the codes, to see whether the codes cover the actual structural resistance of a frame including a leaning column. It was found that using the buckling length in the design rules determined with the nomogram in the Dutch National Annex NB.NA (the method that is most used in practice) gave results that were way off from the most realistic results, namely the buckling length determined with the FE-model. For

that reason, literature is studied where the leaning column effect was included in the buckling length. This gave promising results since all the results were safe and close to the most realistic approach.

To check whether the results were safe for all cases, a large range of leaning column problems was studied in a parametric study. The frame in Figure 1 was extended with a larger leaning column load and a larger horizontal load to increase the range of the leaning column problems. Figure 1 is also shortened to a frame without a leaning column for validation reasons. These new frame configurations are analyzed with variable parameters C and $\bar{\lambda}$, where C is the stiffness ratio and $\bar{\lambda}$ the slenderness ratio. Combining all the frame configurations and variable parameters resulted in 105 different frames that are analyzed, where 30 configurations were without a leaning column and 75 were with a leaning column.

For the 75 frames with leaning columns, it has been found that both the NEN 6771 and the current EN 1993-1-1 design rules do not give safe results when the slenderness ratio of the frames is increasing. This has led to the use of the most accurate method where the leaning column effect is included in the buckling length, namely the Yura approach. The Yura approach is called after the author J.A. Yura and makes use of simple equilibrium combined with the Euler buckling equation. For all considered frame configurations the Yura approach gives safe results, it is very accurate, and besides that, very practical to use.

TABLE OF CONTENTS

<i>ACKNOWLEDGMENTS</i>	1
<i>SUMMARY</i>	2
<i>NOMENCLATURE</i>	7
1 INTRODUCTION	10
1.1 MOTIVATION AND PROBLEM DEFINITION	10
1.1.1 The leaning column effect	10
1.1.2 Design rules according to the codes	11
1.2 RESEARCH OBJECTIVE	18
1.3 RESEARCH APPROACH	18
1.4 RESEARCH SCOPE	19
2 MODIFIED DESIGN RULE EN 1993-1-1	21
2.1 DEVELOPMENT MODIFIED DESIGN RULE EN 1993-1-1	21
3 STRUCTURAL STABILITY OF FRAMES	24
3.1 ANALYSIS METHODS	24
3.2 IMPERFECTIONS	25
3.3 ANALYSIS METHODS USED AND THEIR CORRESPONDING IMPERFECTIONS	26
3.3.1 GMNIA I according to EN 1993-1-1 Clause 5.3.2(3)	27
3.3.2 GMNIA II according to EN 1993-1-1 clause 5.3.2(11)	28
3.3.3 GMNIA III according to Vogel et al.	29
3.3.4 GMNIA IV according to Shayan et al. and Vogel et al.	32
4 FINITE ELEMENT MODEL	34
4.1 ELEMENT TYPE	34
4.2 RELEVANT ANALYSE METHODS OF ABAQUS	34
4.2.1 Static, general	34
4.2.2 Static, Riks	35
4.2.3 Linear perturbation buckling	35
4.3 SHELL ELEMENT MODEL	35
4.3.1 Parts of the frame	35
4.3.2 Interaction of parts	37
4.3.3 Supports	39
4.3.4 Cross-section	40
4.3.5 Mesh	41
4.4 Comparison	41
5 VALIDATION STUDY SWAY FRAME	42
5.1 INTRODUCTION	42
5.2 ANALYSIS METHODS AND IMPERFECTIONS	42
5.3 MESH REFINEMENT STUDY AND THE CORRESPONDING RESIDUAL STRESSES	43
5.4 NUMERICAL RESULTS OF THE VARIOUS ANALYSES - VALIDATION STUDY	45

5.4.1	LBA	45
5.4.2	GMNIA I according to EN 1993-1-1 clause 5.3.2(3) (in-plane)	46
5.4.3	GMNIA II according to EN 1993-1-1 clause 5.3.2(11)	47
5.4.4	GMNIA IIa according to EN 1993-1-1 clause 5.3.2(3).....	48
5.4.5	GMNIA IIb according to NEN 6771	48
5.4.6	GMNIA III according to Vogel et al.	50
5.4.7	GMNIA IV according to Shayan et al. and Vogel et al.	51
5.5	COMPARISON OF RESULTS	53
6	<i>CASE STUDY SWAY FRAME INCLUDING LEANING COLUMN</i>	55
6.1	INTRODUCTION	55
6.2	ANALYSIS METHOD AND IMPERFECTIONS.....	56
6.3	MESH REFINEMENT STUDY AND THE CORRESPONDING RESIDUAL STRESSES.....	57
6.4	NUMERICAL RESULTS OF THE VARIOUS ANALYSES.....	59
6.4.1	LBA	59
6.4.2	GMNIA I according to EN 1993-1-1 Clause 5.3.2(3).....	60
6.4.3	GMNIA II according to EN 1993-1-1 clause 5.3.2(11)	61
6.4.4	GMNIA III according to Vogel et al.	62
6.4.5	GMNIA IV according to Shayan et al. and Vogel et al.	63
6.5	ULTIMATE RESISTANCE LOADS ACCORDING TO THE CODES	65
6.5.1	NEN 6771	65
6.5.2	EN 1993-1-1	66
6.5.3	Modified EC1991-1-1.....	67
6.6	COMPARISON OF THE RESULTS	68
6.6.1	Differences and similarities.....	68
6.7	INFLUENCE OF THE BUCKLING LENGTH DETERMINATION METHOD.....	70
6.8	CONCLUSIONS.....	78
7	<i>PARAMETRIC STUDY</i>	80
7.1	INTRODUCTION	80
7.2	PARAMETERS.....	80
7.3	FRAME CONFIGURATION.....	81
7.3.1	Frame A	81
7.3.2	Frame A-NL.....	82
7.3.3	Frame A-V.....	83
7.3.4	Frame A-H, A-NLH, and A-VH	84
7.3.5	Frame B	86
7.4	RESULTS.....	87
7.4.1	Frame A, A-NL, A-V, and B.....	87
7.4.2	Frame A-H, A-NLH, and A-VH	88
7.5	CONCLUSIONS.....	90
7.5.1	Frame A, A-NL, A-V, and B.....	90
7.5.2	Frames A-H, A-NLH, and A-VH.....	92
7.5.3	Differences between GMNIA III and GMNIA IV	93
8	<i>CONCLUSIONS AND RECOMMENDATIONS</i>	94
8.1	Conclusions	94

8.2 Recommendations..... 96

9 *BIBLIOGRAPHY* 97

APPENDIX A – LITERATURE AND INPUTS FOR THE DESIGN CODES.....

APPENDIX B – INPUT FEM MODEL PARAMETRIC STUDY.....

APPENDIX C – INPUT and OUTPUT FOR DESIGN RULES.....

APPENDIX D – SCRIPT

NOMENCLATURE

LIST OF ABBRAVIATIONS

AF	= Applied Force
DR	= Design-Resistance
EC	= EuroCode
FE	= Finite-Element
FEM	= Finite-Element-Method
GMNIA	= Geometrical and Material Non-linear Analysis with Imperfections
LA	= Linear elastic Analysis
LBA	= Linear elastic Bifurcation Analysis
LPF	= Load Proportionality Factor
LTB	= Lateral Torsional Buckling
NEN	= NEderlandse Norm
NMD	= Normalized Matrix of Displacements
RF	= Reaction Force
U.C.	= Unity Check

LIST OF SYMBOLS

"symbol" _{x,y,z}	= direction indication for each symbol, X-, Y- or Z-direction
α	= imperfection factor, which depends on the type of the cross-section, the thickness, the yield strength, and the buckling plane
α_{cr}	= amplification factor by which the design loads would have to be increased to cause elastic instability
$\alpha_{GMNIA,i}$	= load proportionality factor, where $GMNIA,i$ stand for the corresponding GMNIA
α_h	= reduction factor for height h applicable to columns
α_k	= imperfection factor, in NEN 6771 format
α_m	= reduction factor for the number of columns in a row
$\alpha_{ult,k}$	= minimum load amplifier of the design loads to reach the characteristic resistance of the most critical cross-section.
β	= modified length factor to increase the length of the structure to obtain the critical buckling length
$\beta_{nomogram}$	= modified length factor that does not take into account leaning column load, determined with the nomogram
β_{Yura}	= modified length factor that takes into account the leaning column load, determined with the Yura approach
γ_1	= partial material factor
δ	= (bow) deflection at the middle of the member
Δ	= lateral side-sway deformation of the frame
η_{cr}	= shape of the elastic critical buckling mode
η_{init}	= amplitude of the elastic critical buckling mode
θ_i	= angle
$\bar{\lambda}$	= relative slenderness ratio
λ_0	= plateau value for the buckling curve (0.2)
λ_e	= slenderness of a member where the Euler buckling stress is equal to $f_{y,d}$ in NEN 6771 format
λ_n	= value that represents the nomogram of the Dutch National Annex NB.NA
λ_{rel}	= relative slenderness ratio in NEN 6771 format
$\lambda_{y,z}$	= slenderness of a member relating to buckling of the relevant axis, in NEN 6771 format
μ	= correction factor to take the boundary condition at the other end of the supporting beam into account
ν	= Poisson's ratio in the elastic stage
ϕ	= slope of the structure due to global initial sway imperfection
ϕ_0	= basic value for global initial sway imperfection
χ	= buckling reduction factor
a	= leverarm
A	= area of the cross-section
A_j	= imperfection amplitude for each buckling mode
b	= width
b_1	= width of the stabilizing frame
b_2	= width of the leaning beam that connects the leaning column to the stabilizing frame
C	= stiffness ratio
C_i	= flexibility parameter, where i indicates A or B (the two member ends of the considered column)

C_m	= equivalent uniform moment factor
e_0	= initial bow imperfection of the column also called the Ayrton-Perry format
$e_{0,\delta}$	= bow imperfection, including additional deflection due to 2nd order behavior
E	= modulus of elasticity
$EI \eta_{cr}'' _{max}$	= maximum bending moment due to η_{cr} at the critical cross-section
$EI_{Q=10}$	= bending stiffness (EI) increased with a factor of 10 to prevent the wrong failure mode
f_u	= ultimate strength
f_y	= yield strength
$f_{y,d}$	= design value of the yield strength
$f_{y,d,Q=10}$	= design value of the yield strength increased with a factor of 10 to prevent the wrong failure mode
F	= applied vertical load
F_{cr}	= critical Euler buckling load
F_{lean}	= additional vertical design load that the stabilizing column has to carry
F_{shayan}	= statistically determined single factor
F_{tot}	= total vertical design load that the stabilizing column has to carry
F_{ult}	= ultimate load that the frame can carry
$F_{ult,GMNIA,i}$	= ultimate load that the frame can carry obtained with FEM, where GMNIA,i stands for the corresponding GMNIA
G_i	= flexibility parameter in American stand format, where i indicates A or B (the two member ends of the considered column)
h	= height of the structure
H	= applied horizontal load
i_i	= Surface momentum regarding the corresponding axis, in NEN 6771 format
I_i	= moment of inertia, where i indicates the axis direction
I_{bm}	= moment of inertia of the beam
I_{cln}	= moment of inertia of the column
k	= interaction factor depending on the relevant instability and plasticity phenomena
k_1	= reduction factor for columns longer than 5 m, in NEN 6771 format
k_2	= factor that takes into account the number of columns to be supported under compression or compression and bending, in NEN 6771 format
K	= modified length factor in American standard format
K_{exact}	= exact modified length factor in American standard format
K_n	= modified length factor that takes into account the leaning column load, in American standard format
K_o	= modified length factor that does take into account the leaning column load, in American standard format
L	= length of the member
L_{bm}	= length of the beam
L_{cln}	= length of the column
L_{cr}	= critical buckling length
m	= number of columns in a row
M	= bending moment
M_{el}	= elastic bending moment resistance of the cross-section
M_{Ed}	= design value of the maximum bending moment along the member
$M_{0,Ed}$	= design value of the first-order bending moment
$M_{II,Ed}$	= design value of the 2nd order bending moment
$M_{Ed,max}^{II}$	= maximum design value of the 2 nd order bending moment
M_{pl}	= plastic bending moment resistance of the cross-section
M_{Rd}	= design value of the elastic or plastic moment resistance of the column, depending on the cross-section class
M_{Rk}	= characteristic moment resistance of the cross-section
$M_{u,d}$	= design value of the bending moment resistance of the column in NEN 6771 format
n	= amplification factor in NEN 6771 format
n_s	= highest number of floors of the frame, in NEN 6771 format
$N_{c,u,d}$	= design value of the normal force resistance of the column in NEN 6771 format
N_{Ed}	= design value of the axial load
N_{pl}	= plastic resistance to normal force of the cross-section
N_{Rd}	= design value of the plastic load resistance of the column cross-section
N_{Rk}	= characteristic resistance to normal force of the cross-section
N_{ult}	= ultimate design resistance load
$N_{ult,DR,ij}$	= ultimate design resistance load, where i stands for the corresponding design code and j for the corresponding method for determining L_{cr}
P	= distance from the central point of the arc to the chord length
$P-\Delta$	= 2 nd order behavior of the total structure

$P-\delta$	= 2 nd order behavior of a single member along the length
P_j	= proportion factor of each buckling mode
Q	= applied leaning column load
r	= radius of the arc of the buckling shape
W_i	= section modulus where i stands for elastic or plastic
U	= lateral displacement obtained from Abaqus
x	= distance from the central point of the arc to the actual sway imperfection along L_{cr}
y	= distance from the central point of the arc to the actual sway imperfection perpendicular to the L_{cr} direction

1 INTRODUCTION

1.1 MOTIVATION AND PROBLEM DEFINITION

1.1.1 The leaning column effect

In design rules, the stability of a column is often considered as a function of an individual column. However, the stability of a column is actually a function of all of the columns in the story and therefore a system problem, not only an individual member problem [1]. The so-called 2nd order effects will introduce additional loads on an individual member induced by other members. The 2nd order effects of a structure can be sub-divided into a P- Δ and P- δ effect. P- Δ refers to the 2nd order behavior of the total structure, and P- δ refers to the 2nd order behavior of a single member along the length. This is illustrated in Figure 2, where it can be seen that P- Δ concerns the lateral deflection of the frame and P- δ the deflection of the member. P- Δ effects require an accurate determination of the buckling modes and the corresponding buckling lengths.

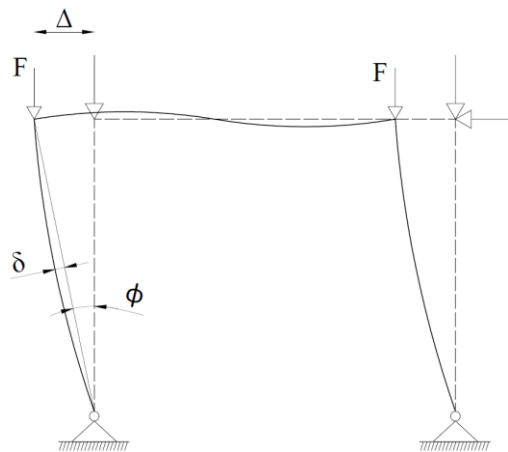


Figure 2: P- Δ and P- δ effect for a sway frame [2]

The above can be translated into a frame with a leaning column. Consider an unbraced sway frame with pin-ended supported columns as shown in Figure 3. The stabilizing frame takes care of the stability of the leaning column, so the leaning column does not participate in the lateral resistance of the structure. When considering the P- Δ effects for the frame of Figure 3, one can see that in the deformed situation, a horizontal force due to leaning will act on the frame coming from the leaning column, illustrated in Figure 4. The stabilizing frame also has to stabilize the leaning column, resulting in a greater 2nd order effect.

Section 1.1.2.1 will show that the NEN 6771 tried to consider the leaning column effect in a system problem, where they consider additional loads on the stabilizing columns. These additional loads represent the loads from the leaning column effect. In Section 1.1.2.2 it will be seen that the leaning column effect is not present in the Eurocode series, since the EN 1993-1-1 considers the criterium as an individual problem.

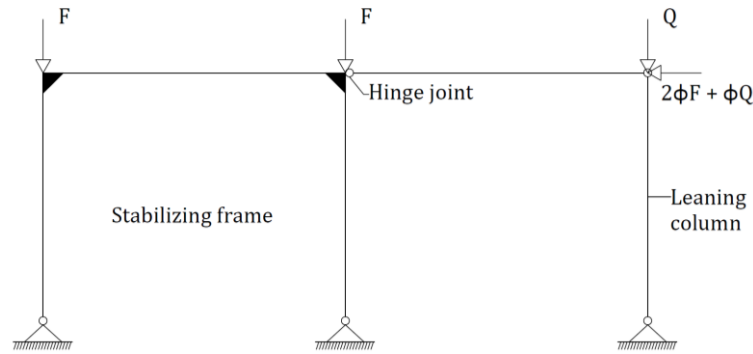


Figure 3: Frame configuration with a stabilizing frame and a leaning column

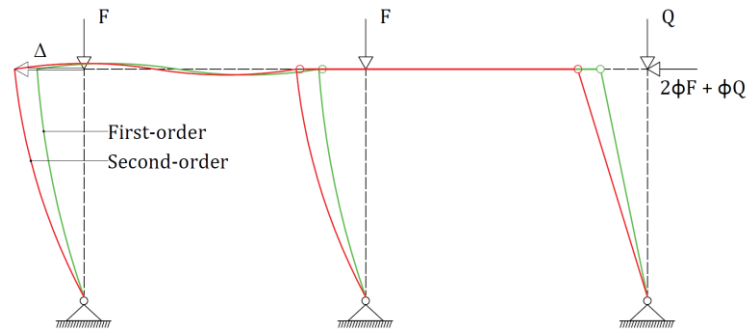


Figure 4: P-Δ effect for a sway frame with a leaning column

1.1.2 Design rules according to the codes

1.1.2.1 Design rules for member stability according to the NEN 6771

In the early days, the NEN 6770 and NEN 6771 provided design rules for steel structures in The Netherlands. Design rules about the stability of columns loaded in compression and bending could be found in both standards, but the NEN 6771 offers the clearest design rules. The assessment criteria for a column loaded in compression and bending is given as the following:

$$\frac{N_{c;s;d}}{N_{c;u;d}} + \frac{n}{n-1} \cdot \frac{F_{tot;s;d} \cdot e^* + M_{equ;s;d}}{M_{u;d}} \leq 1 \quad (1)$$

For comparison reasons, equation (1), the design rule of the NEN 6771, is given with the abbreviations used by the EN 1993-1-1, given as the following:

$$\frac{N_{Ed}}{N_{Rd}} + \frac{\alpha_{cr}}{\alpha_{cr} - 1} \cdot \frac{F_{tot} \cdot e_0 + M_{Ed}}{M_{Rd}} \leq 1 \quad (2)$$

Where:

N_{Ed} = design value of the axial load ($N_{c;s;d}$ in NEN 6771 format);

N_{Rd} = design value of the plastic load resistance of the column cross-section ($N_{c;u;d}$ in NEN 6771 format);

α_{cr} = amplification factor by which the design loads would have to be increased to cause elastic instability (n in NEN 6771 format);

F_{tot} = total vertical load on one considered column, including a part of the vertical load on the leaning column(s) ($F_{tot;s;d}$ in NEN 6771 format);

e_0 = initial bow imperfection of the column, δ in Figure 2 (e^* in NEN 6771 format);

M_{Ed} = design value of the maximum bending moment along the member ($M_{equ;s;d}$ in NEN 6771 format);

M_{Rd} = design value of the elastic or plastic moment resistance of the column, depending on the cross-section class ($M_{u;d}$ in NEN 6771 format).

Where e_0 in equation (2) follows from the Ayrton-Perry format and is given in equation (3):

$$e_0 = \alpha_k(\lambda_{rel} - \lambda_0) \cdot \frac{W}{A} \quad (3)$$

The derivation of equation (3) will be given in section 1.1.2.2 based on the EN 1993-1-1, but in this section, the values from the NEN 6771 regarding equation (3) will be discussed without further explanation. Value λ_{rel} in equation (3) is the relative slenderness given by $\lambda_{rel} = \lambda_y/\lambda_e$, where $\lambda_y = L_{cr}/i_y$ and $\lambda_e = \pi \cdot \sqrt{E/f_y}$, where $i_y = \sqrt{I/A}$. Values E and f_y are the Young's modulus and yield strength of the material respectively. Values A and I are the area and the moment of inertia of the cross-section respectively. The critical buckling length (L_{cr}) can be determined from the Euler buckling load equation:

$$F_{cr} = \frac{\pi^2 EI}{(\beta \cdot L)^2} = \frac{\pi^2 EI}{L_{cr}^2} \quad (4)$$

Equation (2) is a design rule for a column under compression and bending in a two-dimensional frame. Since it is a two-dimensional case, lateral- and flexural torsional buckling are not considered. Equation part N_{Ed}/N_{Rd} represents the plastic member resistance unity-check due to normal force in the column, without considering buckling. The amplification factor $\alpha_{cr}/(\alpha_{cr} - 1)$ takes into account the second-order effects, where $\alpha_{cr} = F_{cr}/N_{Ed}$, where F_{cr} is the elastic critical Euler buckling load as given by equation (4). Criterium $(F_{tot} \cdot e_0 + M_{Ed})/M_{Rd}$ represents the ratio between the elastic or plastic bending moment resistance of the column and the acting bending moments due to imperfections, the leaning column effect, and the loads on the column.

The last part of the equation, the bending moment criterium, and second-order effects can be explained by considering a single column and the corresponding P- δ effect, as illustrated in Figure 5. The column in Figure 5 shows an imperfect column loaded in compression by vertical load F . In the loaded state, the first-order bending moment at equilibrium is equal to $M_{0,Ed} = F \cdot e_0$, where value e_0 is given as the Ayrton Perry format, which describes the real behavior of a column, including buckling. However, due to deflection e_0 , an additional non-linear lateral deflection ' $e_{0,\delta}$ ' will occur and the bending moment at equilibrium will increase to $M_{II,Ed} = F(e_0 + e_{0,\delta})$. This is called the second-order effect and can also be derived from multiplying the first-order bending moment with the amplification factor, leading to $M_{II,Ed} = M_{0,Ed} \cdot \frac{\alpha_{cr}}{\alpha_{cr}-1}$. Combining these equations leads to equation (5):

$$F \cdot (e_0 + e_{0,\delta}) = F \cdot e_0 \cdot \left(\frac{\alpha_{cr}}{\alpha_{cr} - 1} \right) \text{ with } \alpha_{cr} = \frac{F_{cr}}{N_{Ed}} \quad (5)$$

The second term of equation (5) contains a big part of the last term of equation (2), meaning that it takes into account the second-order effects as just described.

As discussed in section 1.1.1, the P- Δ effect should also be considered to include the leaning column effect. In the NEN 6771, the effect of the leaning column is included in the load side of the equation, namely the value F_{tot} . The NEN 6771 offers solutions on how to include the effect of the leaning column in F_{tot} , and these solutions that are based on equilibrium are dependent on the mechanical scheme of the frame. Figure 6 shows how the NEN 6771 considers the total vertical load (F_{tot}) for the unbraced sway frame in Figure 3. In this case, the vertical load Q from the leaning

column is divided over the stabilizing columns of the frame, resulting in $3/2 N_{Ed}$ per stabilizing column. However, the background of the accuracy of this design rule is unknown.

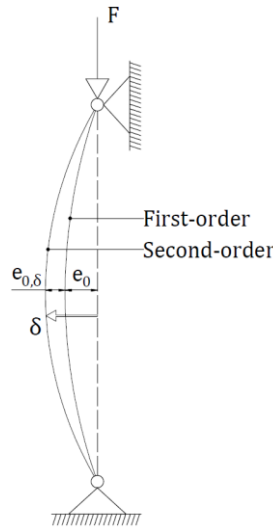


Figure 5: P-δ effect due to vertical loading on an imperfect column

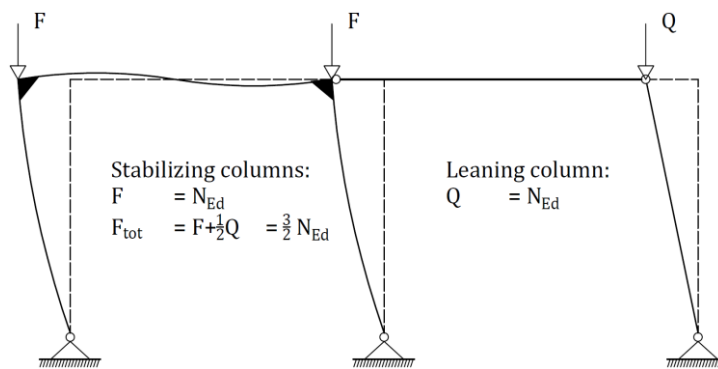


Figure 6: Derivation of F_{tot} for an unbraced sway frame [3]

1.1.2.2 Design rules for member stability according to the Eurocode

In 2012, the Eurocode (EC) was adopted in The Netherlands as a replacement for the NEN standards. The Eurocode is a European standard to assess the structural performance and safety of all possible building structures. The EC EN 1993-1-1 is the basic part containing the design rules for steel structures. The design rule for columns loaded in compression and bending according to the EN 1993-1-1 is given as the following:

$$\frac{N_{Ed}}{\chi_y \cdot N_{Rd}} + k_{yy} \frac{M_{y,Ed} + \Delta M_{y,Ed}}{\chi_{LT} \cdot M_{y,Rd}} + k_{yz} \frac{M_{z,Ed} + \Delta M_{z,Ed}}{\chi_{LT} \cdot M_{z,Rd}} \leq 1 \tag{6}$$

Equation (6) is the three-dimensional design rule for columns loaded in the Y-direction. In this research, only a two-dimensional case will be considered en therefor equation (6) will be reduced to:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \frac{M_{Ed} + \Delta M}{\chi_{LT} M_{Rd}} \leq 1 \tag{7}$$

Where:

N_{Ed} = design value of the axial load;

- N_{Rd} = design value of the plastic load resistance of the column cross-section;
 χ = buckling reduction factor;
 k = interaction factors depending on the relevant instability and plasticity phenomena [2];
 M_{Ed} = design value of the maximum bending moment along the member;
 ΔM = bending moments due to the shift of the center of gravity in case of local buckling
 M_{Rd} = design value of the elastic or plastic moment resistance of the column, depending on the cross-section class.
 χ_{LT} = Lateral torsional buckling reduction factor;

Since local buckling and any out-of-plane behavior is not of interest, values χ_{LT} and ΔM in equation (7) can be left out. Lateral torsional buckling reduction factor χ_{LT} is left out because a two-dimensional case is considered and any out-of-plane (lateral torsional buckling) deformations in the FE-model will not be allowed. The reason for letting out bending moment ΔM is that profile classes 1, 2, and 3 will not be considered to omit local buckling. According to Table 3 of Annex A, this leads to $\Delta M=0$. This will reduce equation (7) to:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \frac{M_{Ed}}{M_{Rd}} \leq 1 \quad (8)$$

Equation (8) is valid for a column under compression and bending in a two-dimensional frame. In this case of this research, the lateral-torsional buckling reduction factor is left out, since any out-of-plane deformations in the Finite Element Model (FE-Model) will not be allowed. The term $N_{Ed}/\chi \cdot N_{Rd}$ presents the member resistance criteria due to normal force in the column, including buckling, where N_{Rd} is given by $N_{Rd} = A \cdot f_y/\gamma_1$, and where γ_1 is the partial resistance factor. For the second term the difference between equation (2) from the NEN 6771, and equation (8) from the EN 1993-1-1, can be spotted in value k .

The value k represents the interaction factor that depends on the relevant instability and plasticity phenomena [2]. The value of k can be derived from two different methods. Back in the days when the EN 1993-1-1 was written, two different teams of researchers treated the field of beam-columns. This has led to two different methods that are both included in the EN 1993-1-1 as Annex A (method 1) and Annex B (method 2). Both methods consider the most complex behavior of a member subjected to compression and biaxial bending, including all possible interactions and non-linear effects [4]. Boissonade et al. [4] state that in method 1 all the influences of material and geometrical nonlinearities and interactions between loading components are reflected by separate factors. Boissonade et al. [4] also state that, in contrast, method 2 uses a reduced number of such factors as a result of the globalization of several effects and calibration of the latter based on extensive numerical solutions. For further background information about the design rule of the Eurocode see chapter 2 of appendix A.

Annex B (method 2) of the Eurocode is used to make a comparison with the content of the NEN 6771. Interaction factor k will be derived for method 2 in the theoretical form. The starting point of the derivation is given in equation (9), which is the criterium for a column loaded in compression and bending [4]:

$$\frac{N_{Ed}}{N_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{C_m \cdot M_{Ed} + N_{Ed} \cdot e_0}{M_{Rk}} \leq 1 \quad (9)$$

Equation (9) makes use of a different notation of the amplification factor. The first step of the derivation is the expression for a column under compression and bending. Basically, equation (9) is the same as equation (2) from the NEN 6771. The only difference in equation (9) is the factor

C_m , N_{Rk} , and M_{Rk} . Values N_{Rk} and M_{Rk} are the characteristic axial resistance and the characteristic bending-moment resistance respectively, given by $N_{Rk} = A \cdot f_y$ and $M_{Rk} = W \cdot f_y$. Factor C_m is the equivalent uniform moment factor depending on the corresponding bending moment diagram. The factor C_m is a standard value that can be obtained from the EN 1993-1-1, which indicates where the maximum bending moment can be found along the beam. This is explained in Figure 7, where the maximum bending moments $M_{0,Ed} = F \cdot e_0$ and $M = F \cdot a$ are not acting on the same location of the column. With the factor C_m from the EN 1993-1-1 the maximum acting bending moment on the column, $M_{Ed,max}^{II} = C_m \cdot M$ can be found.

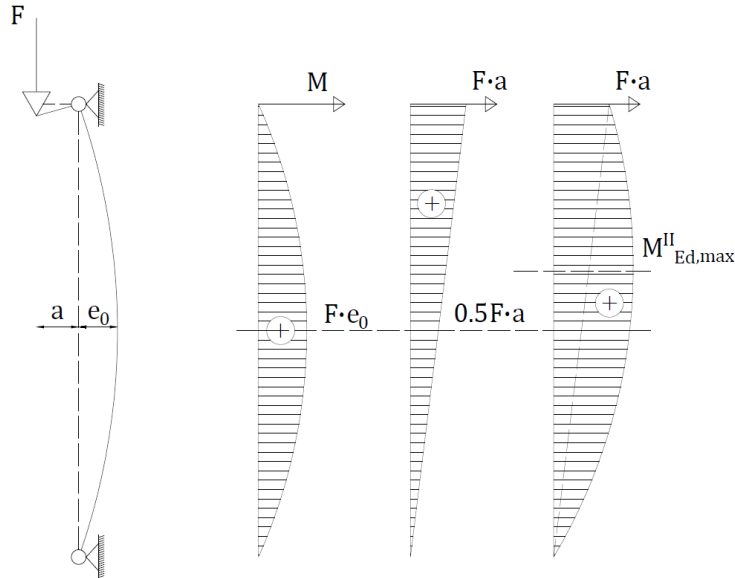


Figure 7: Column with an imperfection loaded on compression and bending, where $M_{Ed,max}$ shifts [5]

Until now, imperfection e_0 had no value. To proceed with the derivation of method 2, this value of imperfection e_0 needs to be included. Therefore e_0 will be numerically derived. This will be done by considering a column only loaded in compression. With the characteristic resistance values considered, the following condition for this column should be met:

$$\frac{N_{Ed}}{N_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{N_{Ed} \cdot e_0}{M_{Rk}} \leq 1 \quad (10)$$

When the axial compression force increases until collapse, N_{Ed} will be equal to $\chi \cdot N_{Rk}$. Substituting this in (10) gives:

$$\frac{\chi \cdot N_{Rk}}{N_{Rk}} + \frac{1}{1 - \frac{\chi \cdot N_{Rk}}{F_{cr}}} \cdot \frac{\chi \cdot N_{Rk} \cdot e_0}{M_{Rk}} = 1 \quad (11)$$

Solving e_0 from equation (11) gives:

$$e_0 = \frac{1 - \chi}{\chi} \cdot \frac{1}{1 - \frac{\chi \cdot N_{Rk}}{F_{cr}}} \cdot \frac{M_{Rk}}{N_{Rk}} \quad (12)$$

For class 1, 2, and 3 the relative slenderness is given by $\bar{\lambda} = \sqrt{N_{Rk}/F_{cr}}$. Substituting $\bar{\lambda}$ in (12) gives the following value for e_0 :

$$e_0 = \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \frac{M_{Rk}}{N_{Rk}} \quad (13)$$

Equation (13) is written in the Ayrton-Perry format and gives a value for e_0 . When proceeding with the derivation of interaction factor k for method 2, equation (13) should be substituted in (9), leading to the following:

$$\frac{N_{Ed}}{N_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{C_m \cdot M_{Ed}}{M_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{N_{Ed}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \leq 1 \quad (14)$$

Multiplying (14) with $1 - N_{Ed}/F_{cr}$ gives:

$$\left(1 - \frac{N_{Ed}}{F_{cr}}\right) \frac{N_{Ed}}{N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \leq \left(1 - \frac{N_{Ed}}{F_{cr}}\right) - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \quad (15)$$

Substituting $\bar{\lambda} = \sqrt{N_{Rk}/F_{cr}}$ in (15) gives:

$$\left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2\right) \frac{N_{Ed}}{N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \leq \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2\right) - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \quad (16)$$

Equation (16) can be rewritten to:

$$\frac{N_{Ed}}{N_{Rk}} - \frac{N_{Ed}^2}{N_{Rk}^2} \cdot \bar{\lambda}^2 + \frac{N_{Ed}}{\chi \cdot N_{Rk}} - \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2 - \frac{N_{Ed}}{N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \cdot \chi \bar{\lambda}^2 - 1 + \frac{N_{Ed}}{N_{Rk}} \bar{\lambda}^2 \leq -\frac{C_m \cdot M_{Ed}}{M_{Rk}} \quad (17)$$

Now some terms in equation (17) cancel each other out and give:

$$-\frac{N_{Ed}^2}{N_{Rk}^2} \bar{\lambda}^2 + \frac{N_{Ed}}{\chi \cdot N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \chi \cdot \bar{\lambda}^2 - 1 \leq -\frac{C_m \cdot M_{Ed}}{M_{Rk}} \quad (18)$$

Equation (18) can be rewritten to:

$$\left(-1 + \frac{N_{Ed}}{\chi \cdot N_{Rk}}\right) \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \chi \cdot \bar{\lambda}^2\right) \leq -\frac{C_m \cdot M_{Ed}}{M_{Rk}} \quad (19)$$

Now the following should be assumed:

$$\frac{1}{k} = \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \chi \cdot \bar{\lambda}^2\right) \quad (20)$$

Substituting (20) in (19) gives:

$$\left(-1 + \frac{N_{Ed}}{\chi \cdot N_{Rk}}\right) \frac{1}{k} + \frac{C_m \cdot M_{Ed}}{M_{Rk}} \leq 0 \quad (21)$$

Multiplying (21) with k gives:

$$\frac{N_{Ed}}{\chi \cdot N_{Rk}} + k \frac{C_m \cdot M_{Ed}}{M_{Rk}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{N_{Ed}}{N_{Rk}} \chi \cdot \bar{\lambda}^2} \quad (22)$$

Combining k in equation (22) with $\bar{\lambda}^2 = N_{Rk}/F_{cr}$ gives:

$$\frac{N_{Ed}}{\chi \cdot N_{Rk}} + k \frac{C_m \cdot M_{Ed}}{M_{Rk}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} \quad (23)$$

When considering cross-section classes 1 and 2, yielding of the full cross-section of the member is allowed. For a cross-section of class 3, the extreme compression fiber can reach its yield strength, but local buckling is liable to prevent the development of the plastic moment resistance [6]. For cross-section classes 1 and 2 this leads to the values $N_{Rk} = N_{pl}$ and $M_{Rk} = M_{pl}$, where N_{pl} and M_{pl} are the plastic axial resistance and the plastic bending-moment resistance respectively. For cross-section class 3 this leads to $N_{Rk} = N_{pl}$ and $M_{Rk} = M_{el}$, where M_{el} is the elastic bending-moment resistance. Substituting these values in equation (23) gives:

$$\frac{N_{Ed}}{\chi \cdot N_{pl}} + k \frac{C_m \cdot M_{Ed}}{M_{el}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} \quad (24)$$

Substituting $N_{pl} = \frac{N_{Rk}}{\gamma_1} = \frac{f_y \cdot A}{\gamma_1} = N_{Rd}$ and $M_{el} = \frac{M_{Rk}}{\gamma_1} = \frac{f_y \cdot W_{el}}{\gamma_1} = M_{Rd}$, where W is the elastic section modulus of the cross-section, in (24) gives:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} \quad (25)$$

The interaction factor k in equation (25) is the theoretical value for method 2 of the EN 1993-1-1. Equation (25) is based on a linear cross-section interaction formulae, while in practice the elastic-plastic behavior for a member under compression and bending is way more complex [5]. This has led to numerical simulations to derive interaction factor k as presented in the EN 1993-1-1. The basis for this formulation is the additional format for the Ayrton-Perry format and is given as $\left(\frac{1}{\chi} - 1\right)(1 - \chi \cdot \bar{\lambda}^2) = \eta\chi = \alpha\chi(\bar{\lambda} - 0.2)$. Tables in the EN 1993-1-1 provide solutions for the interaction factor k , which are based on this value. For a two-dimensional case, a member with profile class 1 or 2, and buckling about the strong axis, this has led to equation (26), which is the final format in the EN 1993-1-1:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \frac{M}{M_{Rd}} \leq 1 \text{ where } k = C_m \left(1 + (\bar{\lambda} - 0.2) \cdot \frac{N}{\chi \cdot N_{Rd}}\right) \leq C_m \left(1 + 0.8 \cdot \frac{N}{\chi \cdot N_{Rd}}\right) \quad (26)$$

From equation (9) it can be seen that the starting point of the format of the EN 1993-1-1 design rule is the same as equation (2) from the NEN 6771 and already taking into account second-order effects. However, from equation (10) to (25) it can be seen that the Eurocode does not explicitly take into account the effect of the leaning column. Both design rules provide design rules for columns loaded in compression and bending but differ in presentation and content. The design rules in the Eurocode makes it hard for authorities to clearly check, what value for the leaning column effect an engineer has accounted for, or to see it has been taken into account at all.

1.2 RESEARCH OBJECTIVE

The research goal is to numerically verify the design rule of the NEN 6771 for the 'leaning column effect' and validate and rewrite the design rule of the EC EN1993-1-1 for a more clear presentation and content in the form of an explicit expression that describes the part of the leaning column effect.

1.3 RESEARCH APPROACH

The numerical verification of the design rules will be done by comparing the ultimate load that the frame can carry to the ultimate resistance load. The ultimate load that the frame can carry will be determined with a geometrical and material non-linear analysis with imperfections (GMNIA) in software Abaqus that reflects reality in the best way possible, resulting in $F_{ult,GMNIA}$. The ultimate resistance load will be determined by increasing load N_{Ed} until the design rule reaches unity and compared to $F_{ult,GMNIA}$. An example for the NEN 6771 is illustrated in Figure 8, where equating the design rule of the NEN 6771 to unity results in the ultimate resistance load, given as $N_{ult,DR,NEN}$.

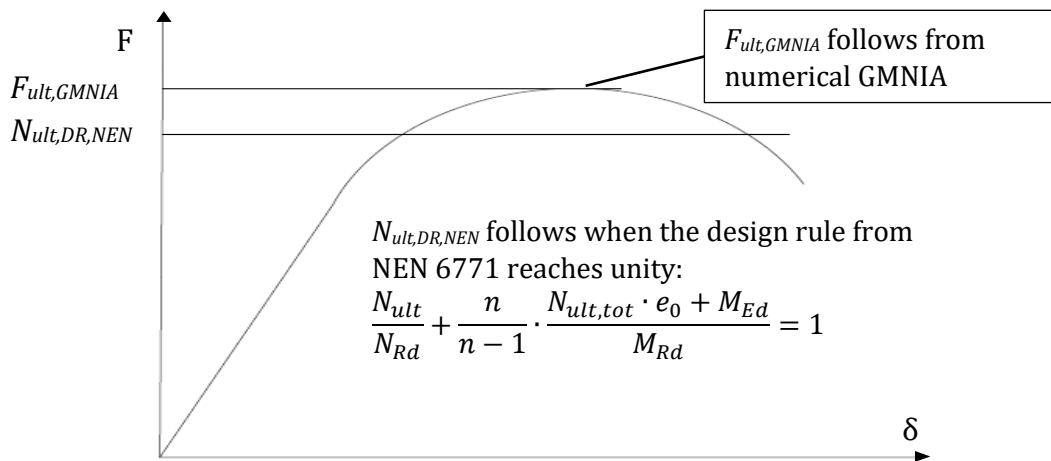


Figure 8: Force-displacement graph where the GMNIA and NEN 6771 will be compared

The design rule of the EN 1993-1-1 will be validated in the same way. The ultimate resistance load $N_{ult,DR,EC}$ will be found after increasing N_{Ed} until unity is reached. This is given by:

$$\frac{N_{ult}}{\chi \cdot N_{Rd}} + k \frac{M_{Ed}}{M_{Rd}} = 1 \quad (27)$$

The design rule of the EN 1993-1-1 will be rewritten with the approach of the NEN 6771, to include the leaning column effect in the load side of the equation, but in the same format as the current EN 1993-1-1 design rule, to make the contribution of the 'leaning column effect' more clear in the EC. When validating the modified design rule of the EN 1993-1-1 ($N_{ult,DR,modEC}$), this should result in the same ultimate resistance load as the NEN 6771.

As a starting point of the research, the frame in Figure 3 will be considered, because it is expected that this configuration has the most impact of the leaning column effect. An FE-Model that can be compared with Figure 3 will be created that reflect reality. To achieve this, a study will be performed about how to create the most realistic model. The analysis method that should determine the ultimate load also should reflect reality. Therefore various analysis methods will be studied. To validate the accuracy of the model, a validation study will be performed over a sway frame without a leaning column, where the results of the literature and the FE-model will be compared. After that, the FE-Model can be used to carry out the analysis and determine $F_{ult,GMNIA}$.

Various schematizations of the frame with a leaning column and variable parameters will be considered. These various schematizations are discussed in section 1.4. The parameters that will be adjusted are the slenderness ratio and the stiffness ratio. The stiffness ratio is defined by $C = \frac{I_{cln}/L_{cln}}{I_{bm}/L_{bm}}$, where I_{cln} and I_{bm} are the second moment of inertia of the column and beam respectively, and where L_{cln} and L_{bm} are the lengths of the column and beam respectively. The stiffness ratio will be pinned on values 1, 1.5, and 2. In ratio C only L_{bm} will be a variable parameter, meaning that for a frame with the same cross-section for column and beam, the length times x1.5 and x2 already lead to stiffness ratios 1.5 and 2. However, theoretically, this ratio difference can also indicate different cross-sections (I_i) between column and beam, which can lead to less stiff structures. The slenderness ratio is defined by $\bar{\lambda} = \sqrt{N_{Rk}/F_{cr}}$, where $F_{cr} = \frac{\pi^2 EI}{L_{cr}^2}$ as given in equation (4) and $N_{Rk} = A \cdot f_y$. For obtaining different slenderness ratios, different cross-sections of the column are used. This makes that the cross-section properties in the equations will be variable, which are A and EI . The smaller the cross-section are over the height, the bigger the slenderness ratio is. Please note that the cross-section of the column also will be used for the beam.

This research focuses on rolled steel cross-sections of classes 1, 2, and 3. The fact that the cross-section is rolled influences the residual stress pattern, which will be used to model the imperfections of the frame. The EN 1993-1-1 classifies steel cross-sections in different classes, namely cross-section classes 1, 2, 3, and 4. Here classes 1 and 2 can develop their full plastic moment resistance and class 3 only its elastic moment resistance. Class 4 only can develop an elastic moment resistance over an effective area, since it is subjected to the effects of local buckling. Since the effect of local buckling does not influence the leaning column effect it is not of interest, and therefore cross-sections of class 4 will not be considered in this research. The leaning column effect is most vulnerable for side-sway modes, and therefore lateral-torsional buckling (LTB) will also not be considered in this research. The reason is that the leaning column effect has the most influence in a two-dimensional case and any out-of-plane deformations in the FE-Model will not be allowed.

This research also will consider different ways of determining the buckling length. The use of FE-software allows the determination of a more accurate value of the buckling length. However, in practice different methods will be used, for example, the nomogram equation, resulting in a higher (unsafely) ultimate load. In the progress, there will be looked for the most accurate way to determine L_{cr} that includes the leaning column effect.

1.4 RESEARCH SCOPE

In this research, a large range of frames will be studied that are the most vulnerable for the leaning column effect. These frame configurations are shown in Figure 9. Frame A is a sway frame with only a small vertical leaning column load and a small horizontal load due to imperfections. Frame A-V is the extended version of A with an increased vertical leaning column load (Q) of x10, where V stands for an increased Vertical load. To avoid that buckling of the leaning column is the governing failure mode, the leaning column is strengthened and stiffened with $10f_{y,d}$, and $10EI$. Frame A-H is also an extended version of A but now with an increased Horizontal load. The horizontal load (H) is based on the grid dimensions of the frame, just as in practice, meaning that the increased horizontal load will not be the same for every frame. Frame A-VH combines the increased horizontal and vertical load. These four frame configurations are all vulnerable to side-sway deformations. For that reason also a stiffer frame configuration is considered, namely frame B. Frame B is a sway frame, but with clamped column ends which reduces the lateral side-sway deformations. The frame is performed with only a small Q and H . To validate the correctness of

the various GMNIA, also frame configurations without a leaning column will be performed, as shown in Figure 10.

All the results for the frames above will be plotted along the slenderness ratio $\bar{\lambda}$ from section 1.3. The frame configurations and the corresponding parameters cover a large range of $\bar{\lambda}$ and will be discussed more in-depth in chapter 7.

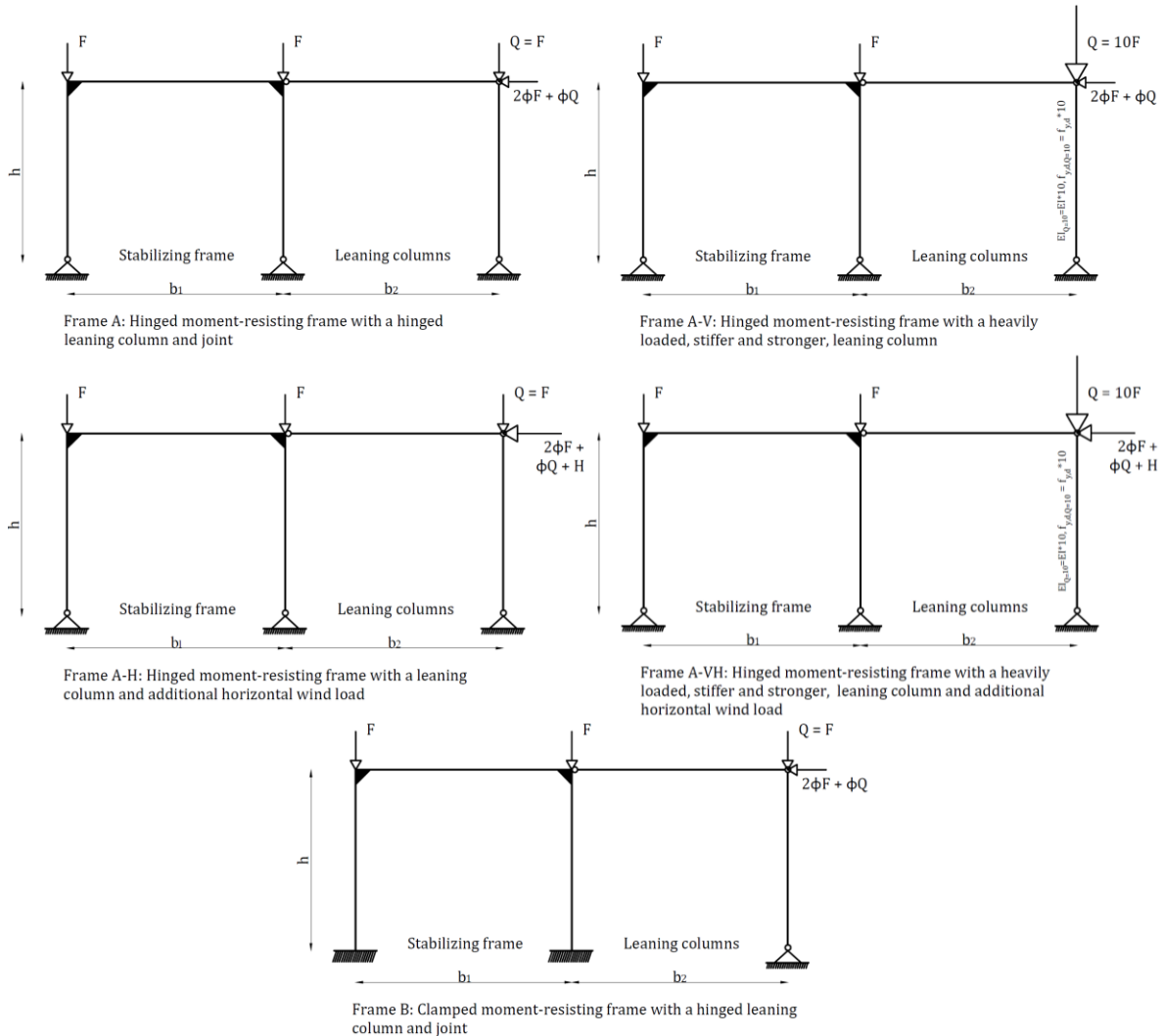


Figure 9: The frame configurations that present the research scope for the leaning column problem

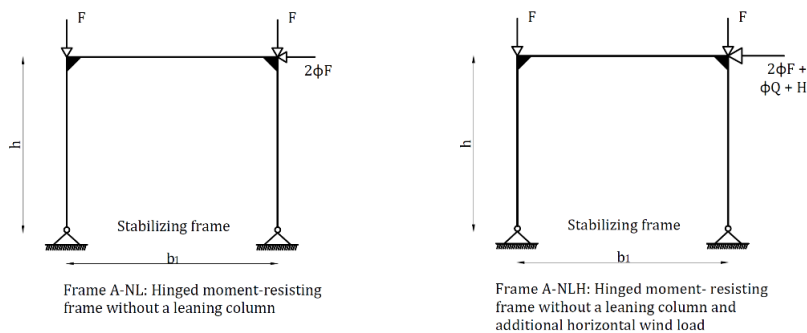


Figure 10: Frame configurations without the leaning column effect to validate the analyses

2 MODIFIED DESIGN RULE EN 1993-1-1

The modified design rule for the EN 1993-1-1, where the leaning column effect is included in the load side of the equation similar to the NEN 6771, can be developed right from the start and will be elaborated on in this chapter. The modified design rule will include the leaning column effect similarly to the NEN 6771. The final format of the modified design rule will be similar to the EN 1993-1-1. In chapter 6, the modified design rule will be used as one of the three ways to determine the ultimate design load. The other two ways are the actual NEN 6771 and EN 1993-1-1 design rules as described in section 1.1.2.

2.1 DEVELOPMENT MODIFIED DESIGN RULE EN 1993-1-1

From section 1.1.2.1 and 1.1.2.2 the two following practical assessment criteria for a column loaded in compression and bending are found (equation (2) and (9) respectively):

Starting point EN 1993-1-1:

$$\frac{N_{Ed}}{N_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{C_m \cdot M_{Ed} + N_{Ed} \cdot e_0}{M_{Rk}} \leq 1 \quad (28)$$

NEN 6771:

$$\frac{N_{Ed}}{N_{Rd}} + \frac{\alpha_{cr}}{\alpha_{cr} - 1} \cdot \frac{F_{tot} \cdot e_0 + M_{Ed}}{M_{Rd}} \leq 1 \quad (29)$$

By implementing the principle of value F_{tot} from equation (28) in equation (29), the leaning column effect can be included in the equation of the EN 1993-1-1. According to Figure 6, F_{tot} is $3/2$ of load F . In the current design rule of the EN 1993-1-1 $1F$ is already assessed by the design rule, meaning that $1/2$ of F is not. By splitting F_{tot} to F and $F_{lean} = 1/2 F$, the leaning column effect can be included in the modified design rule and the current format of the EN 1993-1-1 can be kept. The principle of F_{lean} is shown in Figure 11, which should lead to the same result as the NEN 6771.

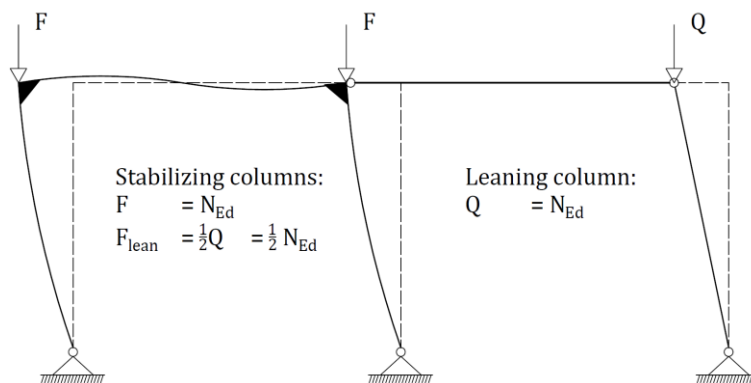


Figure 11: Derivation of F_{lean} for an unbraced sway frame

The derivation for Annex B method 2, as given in section 1.1.2.2, is elaborated again with starting point equation (30):

$$\frac{N_{Ed}}{N_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{C_m \cdot M_{Ed} + N_{Ed} \cdot e_0 + F_{lean} \cdot e_0}{M_{Rk}} \leq 1 \quad (30)$$

Value e_0 is again taken as the Ayrton-Perry format:

$$e_0 = \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \frac{M_{Rk}}{N_{Rk}} \quad (31)$$

Substituting (31) in 30) gives:

$$\begin{aligned} \frac{N_{Ed}}{N_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{C_m \cdot M_{Ed}}{M_{Rk}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{N_{Ed}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \\ + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{F_{lean}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \leq 1 \end{aligned} \quad (32)$$

Multiplying (32) with $1 - N_{Ed}/F_{cr}$ gives:

$$\begin{aligned} \left(1 - \frac{N_{Ed}}{F_{cr}}\right) \frac{N_{Ed}}{N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) + \frac{F_{lean}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \\ \leq \left(1 - \frac{N_{Ed}}{F_{cr}}\right) - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \end{aligned} \quad (33)$$

Substituting $\bar{\lambda} = \sqrt{N_{Rk}/F_{cr}}$ in (33) gives:

$$\begin{aligned} \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2\right) \frac{N_{Ed}}{N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) + \frac{F_{lean}}{N_{Rk}} \left(\frac{1}{\chi} - 1\right) (1 - \chi \cdot \bar{\lambda}^2) \\ \leq \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2\right) - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \end{aligned} \quad (34)$$

Equation (34) can be rewritten to:

$$\begin{aligned} \frac{N_{Ed}}{N_{Rk}} - \frac{N_{Ed}^2}{N_{Rk}^2} \cdot \bar{\lambda}^2 + \frac{N_{Ed}}{\chi \cdot N_{Rk}} - \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2 - \frac{N_{Ed}}{N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \cdot \chi \bar{\lambda}^2 + \frac{F_{lean}}{\chi \cdot N_{Rk}} - \frac{F_{lean}}{N_{Rk}} \cdot \bar{\lambda}^2 \\ - \frac{F_{lean}}{N_{Rk}} + \frac{F_{lean}}{N_{Rk}} \cdot \chi \bar{\lambda}^2 - 1 + \frac{N_{Ed}}{N_{Rk}} \cdot \bar{\lambda}^2 \leq - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \end{aligned} \quad (35)$$

In this case, some of the values with N_{Ed} in equation (35) cancel each other out, but the values with F_{lean} in the equation do not. Therefore, only a part of the equation (35) can be rewritten to:

$$\begin{aligned} - \frac{N_{Ed}^2}{N_{Rk}^2} \bar{\lambda}^2 + \frac{N_{Ed}}{\chi \cdot N_{Rk}} + \frac{N_{Ed}}{N_{Rk}} \chi \cdot \bar{\lambda}^2 + \frac{F_{lean}}{\chi \cdot N_{Rk}} - \frac{F_{lean}}{N_{Rk}} \cdot \bar{\lambda}^2 - \frac{F_{lean}}{N_{Rk}} + \frac{F_{lean}}{N_{Rk}} \cdot \chi \bar{\lambda}^2 \\ - 1 \leq - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \end{aligned} \quad (36)$$

Then equation (36) can be rewritten to:

$$\left(-1 + \frac{N_{Ed}}{\chi \cdot N_{Rk}}\right) \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \chi \cdot \bar{\lambda}^2\right) + \left(\bar{\lambda}^2 - \frac{1}{\chi}\right) (\chi - 1) \frac{F_{lean}}{N_{Rk}} \leq - \frac{C_m \cdot M_{Ed}}{M_{Rk}} \quad (37)$$

Now the following should be assumed:

$$\frac{1}{k} = \left(1 - \frac{N_{Ed}}{N_{Rk}} \cdot \chi \cdot \bar{\lambda}^2\right) \quad (38)$$

Substituting (38) in (37) gives:

$$\left(-1 + \frac{N_{Ed}}{\chi \cdot N_{Rk}}\right) \frac{1}{k} + \left(\bar{\lambda}^2 - \frac{1}{\chi}\right) (\chi - 1) \frac{F_{lean}}{N_{Rk}} + \frac{C_m \cdot M_{Ed}}{M_{Rk}} \leq 0 \quad (39)$$

Multiplying (39) with k gives:

$$\frac{N_{Ed}}{\chi \cdot N_{Rk}} + k \left(\bar{\lambda}^2 - \frac{1}{\chi}\right) (\chi - 1) \frac{F_{lean}}{N_{Rk}} + k \frac{C_m \cdot M_{Ed}}{M_{Rk}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{N_{Ed}}{N_{Rk}} \chi \cdot \bar{\lambda}^2} \quad (40)$$

Combining k in equation (40) with $\bar{\lambda}^2 = N_{Rk}/F_{cr}$ gives:

$$\frac{N_{Ed}}{\chi \cdot N_{Rk}} + k \left(\bar{\lambda}^2 - \frac{1}{\chi}\right) (\chi - 1) \frac{F_{lean}}{N_{Rk}} + k \frac{C_m \cdot M_{Ed}}{M_{Rk}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} \quad (41)$$

As described in section 1.1.2.2, the values for cross-section classes 1 and 2 leads to the values $N_{Rk} = N_{pl}$ and $M_{Rk} = M_{pl}$, where N_{pl} and M_{pl} are the plastic axial resistance and the plastic bending-moment resistance respectively. For cross-section class 3 this leads to $N_{Rk} = N_{pl}$ and $M_{Rk} = M_{el}$, where M_{el} is the elastic bending-moment resistance. Substituting these values in equation (41) gives:

$$\frac{N_{Ed}}{\chi \cdot N_{pl}} + k \left(\bar{\lambda}^2 - \frac{1}{\chi}\right) (\chi - 1) \frac{F_{lean}}{N_{pl}} + k \frac{C_m \cdot M_{Ed}}{M_{el}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} \quad (42)$$

Substituting $N_{pl} = \frac{N_{Rk}}{\gamma_1} = \frac{f_y \cdot A}{\gamma_1} = N_{Rd}$ and $M_{el} = \frac{M_{Rk}}{\gamma_1} = \frac{f_y \cdot W_{el}}{\gamma_1} = M_{Rd}$ in (42) gives:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \left(\bar{\lambda}^2 - \frac{1}{\chi}\right) (\chi - 1) \frac{F_{lean}}{N_{Rd}} + k \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} \quad (43)$$

Where equation (43) is based on a linear cross-section interaction formula and interaction factor k is the theoretical value for method 2 of the EN 1993-1-1. Equation (43) is the format of the modified design rule of the Eurocode and gives the same result as equation (2) from the NEN 6771.

3 STRUCTURAL STABILITY OF FRAMES

To compare the design rules with real structural behavior, the FE analyses should provide sufficient accuracy regarding the structural stability of the frame. This chapter will discuss the types of analysis that have to be considered to study such a frame including the leaning column effect. Non-linearities, or also called imperfections, play a role in the accuracy of the analyses and also will be discussed in this chapter. At the end of this chapter, the considered analyses and the corresponding imperfections that will be used will be listed and discussed.

3.1 ANALYSIS METHODS

Structures carried out in steel are usually slender structures when compared to other materials. Instability phenomena are in most cases present, so it is necessary to verify the global stability of the structure. Due to deformations that slender structures undergo, this verification needs a 2nd order analysis. A global analysis directly accounting for all imperfections (geometrical and material) and all 2nd order effects is the most convenient one, leading to a second-order inelastic analysis. This method is commonly called a Geometrical and Material Non-linear Analysis with Imperfections, the so-called GMNIA [2]. Regarding the input that is needed to carry out the design rules and the GMNIA as will be described in section 3.3, additionally, a linear elastic analysis (LA) and linear elastic bifurcation analysis (LBA) will be performed. The following analysis methods are suitable:

- first-order elastic analysis: In this approach, small deformations without imperfections are assumed. Equilibrium is determined on the undeformed state, so not taking into account additional internal forces due to deformations of the structure. Additionally, superposition is valid and any inelastic behavior of the material is ignored according to Hooke's law, leading to a linear result [1]. This type of analysis applies to a linear elastic analysis (LA).
- elastic buckling analysis: An elastic buckling analysis determines the critical buckling load of the structure. This load can be determined from the eigenvalue of the structure or an iterative process based on the equilibrium in the deformed state. The result of this analysis is only a single value at which the structure buckles and is referred to as the Euler buckling load [1]. The Euler buckling load takes into account linear elastic behavior and small deformations without imperfections. This type of analysis applies to a linear elastic bifurcation analysis (LBA), resulting in α_{cr} .
- second-order elastoplastic analysis: This analysis combines the second-order elastic analysis with the first-order elastic-plastic analysis. This type of analysis takes non-linearity into account and the material behavior is considered as elastoplastic, leading to a realistic behavior reflecting reality. The model takes into account large deformations in such a way that it influences the state of equilibrium [1]. This type of analysis applies to a geometrical and material nonlinear analysis including imperfections (GMNIA).

Figure 12 shows the load-displacement history for the various approaches. It can be seen that the second-order inelastic analysis leads to the lowest ultimate load that the frame can carry compared to other analyses. It should be noted that structures have more than one buckling mode and therefore more than just the first buckling mode should be considered to reflect reality in the best way. Also, for the calculation of the elastic critical load, the structural model that represents the structure should adequately reproduce the elastic behavior, so including the behavior of the joints and initial stiffness.

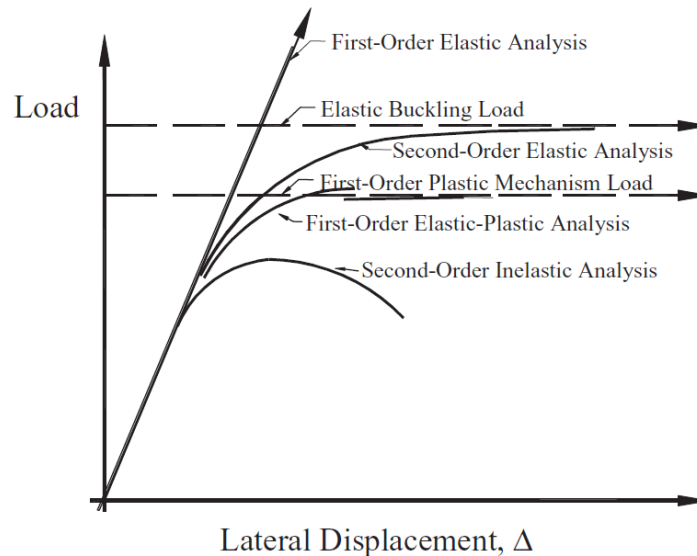


Figure 12: Load-displacement paths [1]

3.2 IMPERFECTIONS

Imperfections, as mentioned in section 1.1, are always present in steel structures, and are responsible for the additional secondary forces that will occur in the structure. There are several types of imperfections:

- Residual stresses (material imperfections);
- Eccentricities in joints (geometrical imperfections);
- Eccentricities of loads (geometrical imperfections);
- lack of verticality, lack of straightness, and lack of flatness in members (geometrical imperfections);
- imperfections leading to torsional effects;
- slip in bolt holes, and similar stiffness effects.

In this research, imperfections will be considered according to the Eurocode and the literature. In the EN 1993-1-1 clause 5.3.1(2) it is stated that in the analysis, the imperfections should be taken into account in the form of equivalent geometric imperfections. This form reflects all imperfections as discussed above. In the case of imperfections based on the literature, as many imperfections as possible will be modeled aiming for the most realistic representation of the frame. The effects of the imperfections that should be taken into account are global imperfections for frames and local imperfections for individual members. [7]

For the imperfections for global analysis of frames, the EN 1993-1-1 clause 5.3.2(1) consider the shape and direction that lead to the governing situation and effects. This makes that the assumed shape of global imperfections and local imperfections may be derived from the direction of the elastic buckling mode. These buckling modes are also dependent on the behavior of the structure. When a structure is sensitive to lateral deformations it is called “sway mode”, and when a structure is not sensitive to lateral deformations it is called “non-sway mode”. This is illustrated in Figure 13. According to the EN 1993-1-1 clause 5.3.2(3), for frames sensitive to buckling in a sway mode, it is allowed to take imperfections into account by considering an equivalent imperfection in the form of an initial sway imperfection and individual bow imperfections of the members.

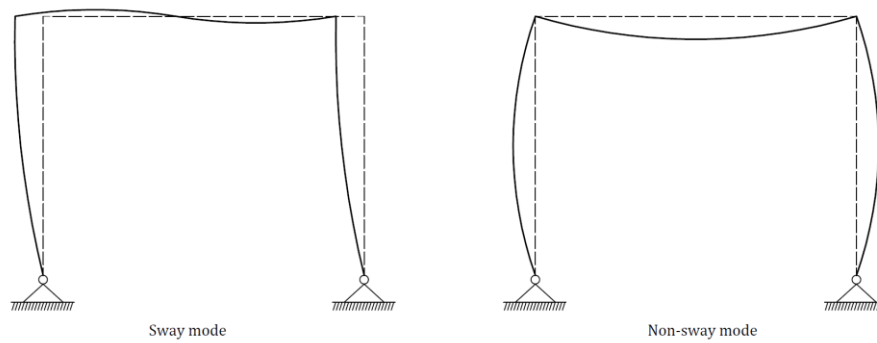


Figure 13: Sway mode frame (left) and non-sway mode frame (right) [2]

The EN 1993-1-1 allows replacing the effects of sway imperfections and local bow imperfections as described above by equivalent horizontal forces, as illustrated in Figure 14.

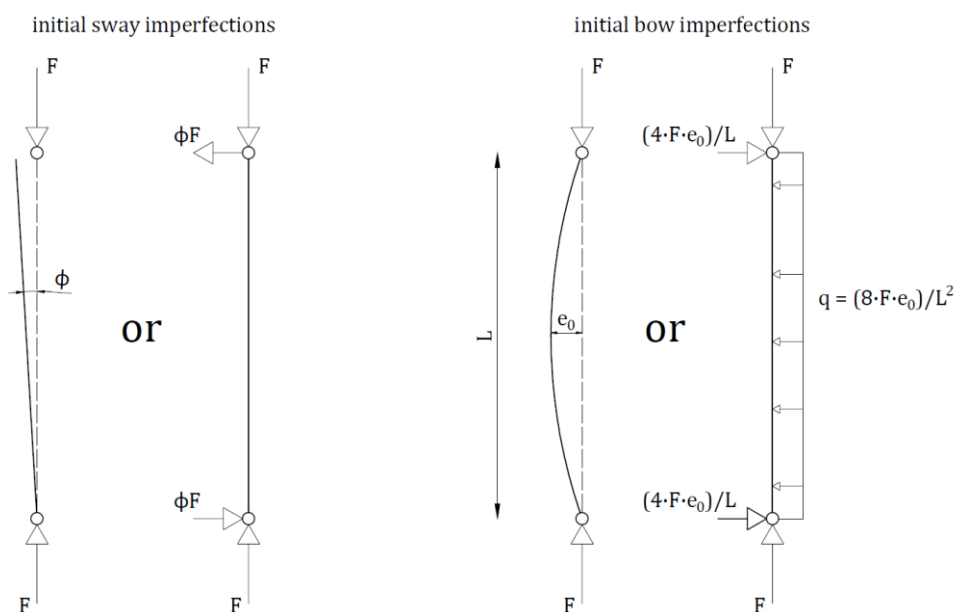


Figure 14: Replacement of initial imperfections by equivalent horizontal forces [7]

Structures are the most sensitive to imperfections like the eigenmode. It is possible to use displacement normalization, which makes use of the actual buckling modes. This technique normalizes the matrix of displacements of the buckling shape and makes it possible to plot the normalized matrix of displacements (NMD) as an imperfection on the structure. This can be performed by the Finite Element software and results in a more realistic behavior of the structure. An additional way to aim for a more realistic model is the implementation of residual stresses. Residual stresses develop due to differential cooling after hot rolling and any other kind of process involving heat and can be included in an FE-Model [2].

3.3 ANALYSIS METHODS USED AND THEIR CORRESPONDING IMPERFECTIONS

To determine the ultimate load that the frame can carry (F_{ult}), various GMNIA with different imperfections will be performed. These GMNIA's aims to accurately determine the ultimate load that the frame can carry. The inputs for the imperfections are based on the EN 1993-1-1 or the literature. To determine the magnitude of the imperfections, in most cases on the forehand an LBA and LA are needed.

3.3.1 GMNIA I according to EN 1993-1-1 Clause 5.3.2(3)

GMNIA I makes use of the equivalent sway and bow imperfections according to the EN 1993-1-1. For the derivation of the global initial sway imperfection, the EN 1993-1-1 clause 5.3.2(3a) [7] refers to Figure 15, indicating a lack of verticality of the structure. The slope ϕ of the imperfect structure is given by:

$$\phi = \phi_0 \cdot \alpha_h \cdot \alpha_m \quad (44)$$

Where ϕ_0 is the basic value of $\phi_0 = 1/200$, α_h is the reduction factor for height h applicable to columns and α_m is the reduction factor for the number of columns in a row, and follow from:

$$\alpha_h = \frac{2}{\sqrt{h}} \quad \text{but} \quad \frac{2}{3} \leq \alpha_h \leq 1 \quad ; \quad \alpha_m = \sqrt{0.5 \left(1 + \frac{1}{m}\right)} \quad (45)$$

Where h is the height of the structure in meters and m is the number of columns in a row, including only those columns which carry a vertical load F not less than 50% of the average value of the column in the vertical plane considered. The lateral displacement can be found by $\Delta = \phi h$.

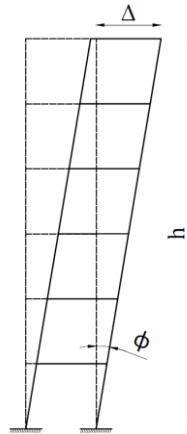


Figure 15: Equivalent sway imperfections [7]

According to EN 1993-1-1 clause 5.3.2(3b) [7], the relative initial local bow imperfections of members for flexural buckling in a bending mode are based on the numerical format of the Ayrton-Perry format, as discussed in section 1.1.2.2. Bow imperfection e_0 is given by equation (46):

$$e_0 = \alpha(\bar{\lambda} - 0.2) \frac{M_{Rk}}{N_{Rk}} \quad (46)$$

Where α is the imperfection factor for the relevant buckling curve. For the value of α see table 1 of appendix A. Value e_0 is the maximum initial lateral displacement over height L , as shown in Figure 16.

The corresponding flow chart of GMNIA I is given in Figure 17, where $\alpha_{ult,k}$ is the minimum force amplifier for the load N_{Ed} to reach the characteristic resistance N_{Rk} of the most axially stressed cross-section without taking buckling into account [7]. The imperfections are induced on the structure by equivalent forces, as illustrated in Figure 14.

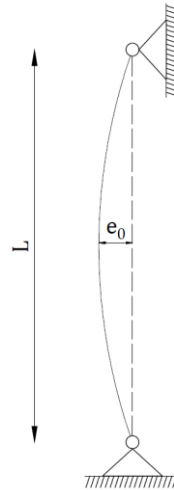


Figure 16: Maximum amplitude of the initial lateral displacement e_0

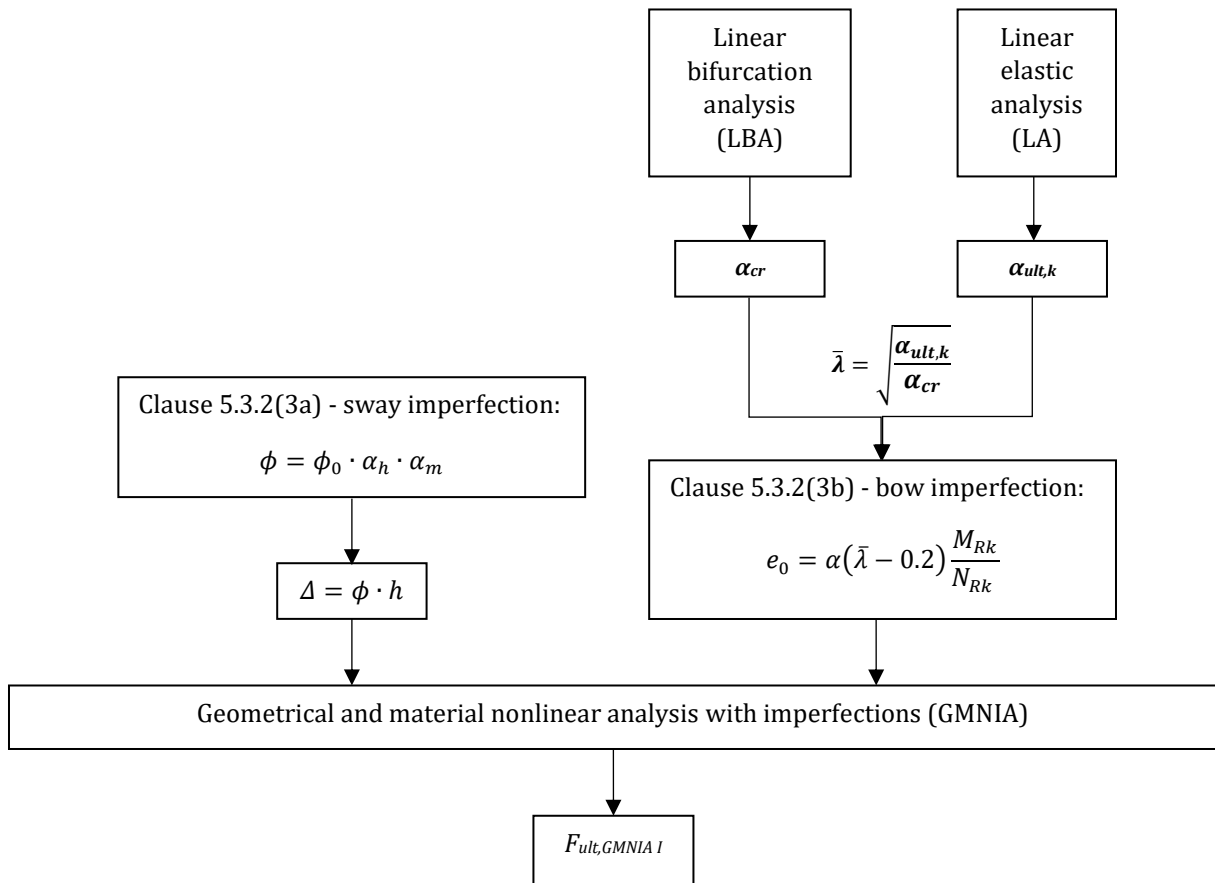


Figure 17: Flow chart of GMNIA I for obtaining $F_{ult,GMNIA I}$

3.3.2 GMNIA II according to EN 1993-1-1 clause 5.3.2(11)

As an alternative for clause 5.3.2(3), one can choose to derive the imperfections with clause 5.3.2(11). This will be used in GMNIA II. Clause 5.3.2(11) induces the imperfections on the structure by scaling the normalized matrix of displacements of the critical eigenmode with equivalent scaling-factor η_{init} . The solutions for clause 5.3.2(11) are given in equation (47) and (48):

$$\eta_{init} = e_0 \cdot \frac{F_{cr}}{EI|\eta_{cr}''|_{max}} \eta_{cr} = \frac{e_0}{\bar{\lambda}^2} \cdot \frac{N_{Rk}}{EI|\eta_{cr}''|_{max}} \eta_{cr} \quad (47)$$

$$e_0 = \alpha(\bar{\lambda} - 0.2) \frac{M_{Rk}}{N_{Rk}} \cdot \frac{1 - \chi^{\bar{\lambda}^2}}{1 - \chi^{\frac{Y_{M1}}{\bar{\lambda}^2}}} \text{ for } \bar{\lambda} > 0.2 \quad (48)$$

Where, according to Comité Européen de Normalisation [7], the abbreviations are the following:

$EI|\eta_{cr}''|_{max}$ = is the bending moment due to η_{cr} at the critical cross-section;

η_{cr} = shape of elastic critical buckling mode.

The corresponding flow chart of GMNIA II is given in Figure 18.

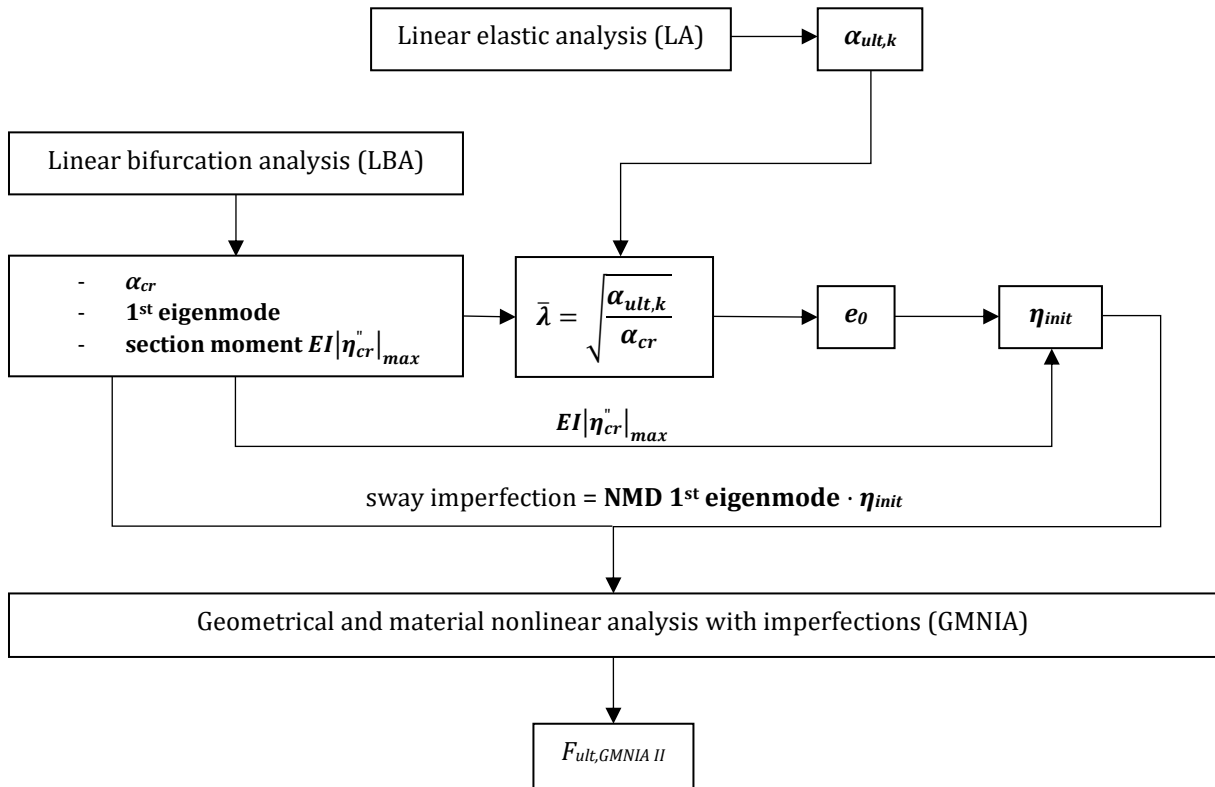


Figure 18: Flow chart of GMNIA II for obtaining $F_{ult,GMNIA II}$

3.3.3 GMNIA III according to Vogel et al.

GMNIA III takes into account geometrical imperfections and residual stresses from the literature, aiming for a more realistic representation of the structure. When one chooses to implement residual stresses, this has some consequences for the GMNIA analyses. The first one is the way of modeling in the software. The equivalent imperfections of the EN 1993-1-1 allow modeling with beam elements since geometrical imperfections do not need many details in the modeling. However, these beam elements do not allow the implementation of residual stresses, leading to the usage of shell elements when residual stresses are considered. These elements need a more detailed approach to modeling and will be discussed in chapter 4. Secondly, the EN 1993-1-1 does not cover the field of analysis with the implementation of residual stresses. This means that all amplitudes of imperfections that have to be included in GMNIA III have to be obtained from the literature.

The European Convention for Constructional Steelwork (ECCS) offers some basic input parameters for the analysis, focussing on the use of computer software. These parameters will lead overall to comparable results. In this research only rolled I-sections will be considered. According to the elastic-plastic analysis that was discussed by Vogel et al. [8], the residual stress pattern that has to be taken into account is shown in Figure 19, and the initial geometrical imperfection in Figure 20.

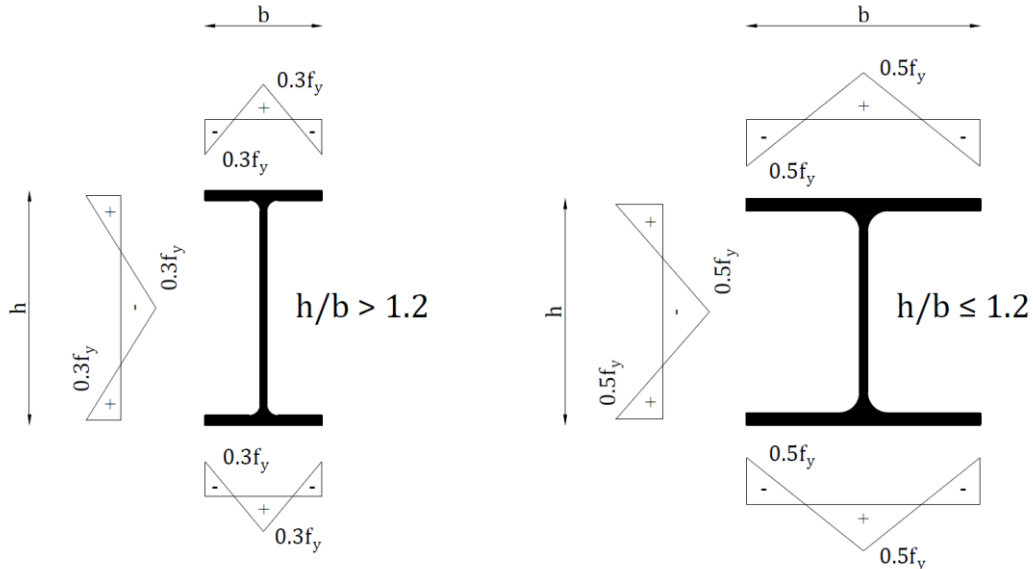


Figure 19: The magnitude of residual stresses for rolled I-shape sections according to Vogel et al. [8]

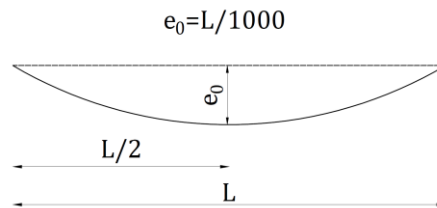


Figure 20: Geometric imperfections for the elastic-plastic analysis according to Vogel et al. [8]

Based on the initial bow imperfection in Figure 20, the sway imperfection can be sized based on the buckling length of the structure. The buckling mode of a structure follows a half-sinus wave, which is the same form as the initial bow imperfection. This means that the lateral sway deflection lays on the line of the buckling mode of the structure, which in this case is equal to $e_0 = L_{cr}/1000$. Figure 21 gives the initial sway imperfections following the corresponding buckling mode for a pin-ended connected frame (left) and a clamped-ended connected frame (right).

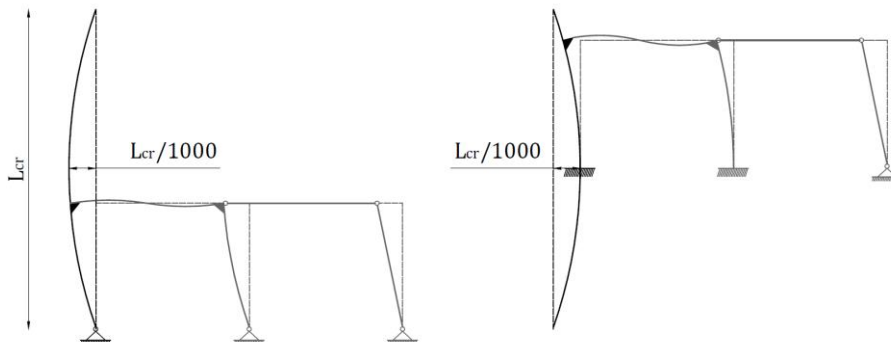


Figure 21: Sway imperfection based on $e_0 = L_{cr}/1000$ for pin-ended frame (left) and clamped-ended frame (right)

Based on equation (4), L_{cr} can be derived from equation (49):

$$L_{cr} = \pi \cdot \sqrt{\frac{EI}{\alpha_{cr} \cdot N_{Ed}}} \tag{49}$$

Where $\alpha_{cr} \cdot N_{Ed}$ represents the critical load F_{cr} . Figure 21 shows that the actual lateral displacement of the frame is not at the height of $L_{cr}/1000$. This is only the case when $L_{cr} = 2L$. To obtain the value at the correct height, geometrical corrections have to be taken. An example of such a geometrical correction is shown in Figure 22.

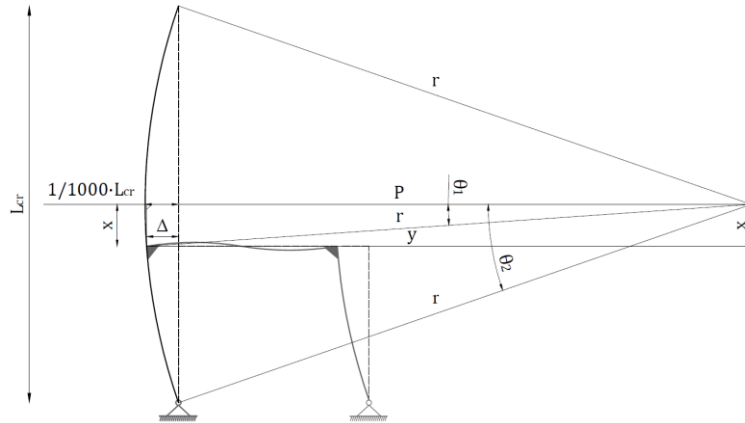


Figure 22: Geometrical correction for the sway imperfection based on Vogel et al. [8]

For GMNIA III the initial geometrical sway imperfections also will be induced on the structure by the normalized matrix displacement of the critical eigenmode obtained by the LBA. The corresponding flow chart is given in Figure 23.

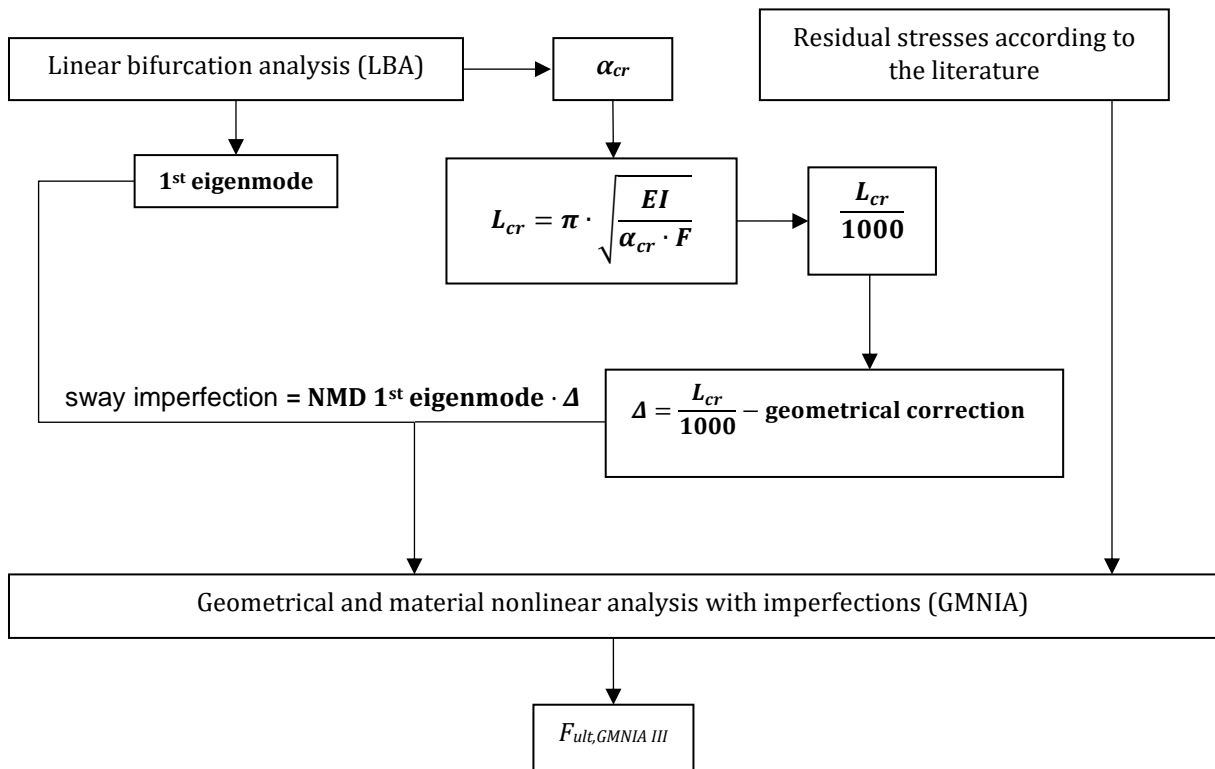


Figure 23: Flow chart of GMNIA III for obtaining $F_{ult,GMNIA III}$

3.3.4 GMNIA IV according to Shayan et al. and Vogel et al.

As stated before, sway and bow imperfections are the initial geometrical imperfections that should be taken into account to cover all influences of the geometrical imperfections or even the material imperfections in the case of the EN 1993-1-1. These initial geometrical imperfections should be implemented in such a way that the worst-case scenario will be obtained which will lead to the ultimate load. However, Shayan et al. [9] state that the worst-case scenario of imperfections may be overly conservative, and that also in a lot of cases it is not clear on forehand what will be the worst-case scenario. For that reason, Shayan et al. [9] researched analyses for steel structural systems and developed a rational method of modeling initial geometric imperfections. With statistical data for initial geometric imperfection, they extended the scaling of elastic buckling mode (EBM). Normally, an EBM is based on the first eigenmode of the structure, which should represent the most critical imperfection geometry (GMNIA II and GMNIA III). However, as a result of plastic deformations, the final failure mode of the frame may be different from the critical elastic buckling mode [9]. In the EBM-method developed by Shayan et al. [9], they consider the initial geometric imperfection as a linear superposition of several scaled buckling modes. From statistical data from other literature, Shayan et al. [9] developed scaling factors for each buckling mode that are presented in Table 1. With one equation, one can calculate the imperfection amplitude for each buckling mode (A_j) that can be implemented in the FE-model. For sway modes the solution is given by equation (50):

$$A_j = P_j \cdot F_{shayan} \cdot h \quad (50)$$

Where P_j is the proportion of each mode, F_{shayan} a single factor, and h the height of the frame. Table 1 offers scaling factors up to ten different buckling modes. However, Shayan et al. [9] show that for unbraced frames, 3 modes already give sufficient accuracy to determine the correct initial geometric imperfection.

Table 1: Proportion of each mode to model initial geometric imperfections for unbraced frames [9]

Number of modes	P_1	P_2	P_3	P_4	P_5	P_6	P_7	P_8	P_9	P_{10}	F_{shayan}
1	1										0.001228
2	0.782	0.218									0.001566
3	0.623	0.161	0.216								0.001838
4	0.522	0.121	0.178	0.178							0.002147
5	0.441	0.103	0.155	0.155	0.143						0.002504
6	0.386	0.090	0.138	0.138	0.120	0.128					0.002817
7	0.350	0.079	0.127	0.127	0.109	0.113	0.098				0.00309
8	0.316	0.071	0.110	0.110	0.103	0.104	0.091	0.093			0.003413
9	0.283	0.064	0.099	0.099	0.091	0.091	0.085	0.089	0.099		0.003782
10	0.266	0.060	0.089	0.089	0.084	0.082	0.077	0.083	0.090	0.077	0.004030

The residual stresses are induced in the same way on the structure as GMNIA III (section 3.3.3). The corresponding flow chart for GMNIA IV is given in Figure 24.

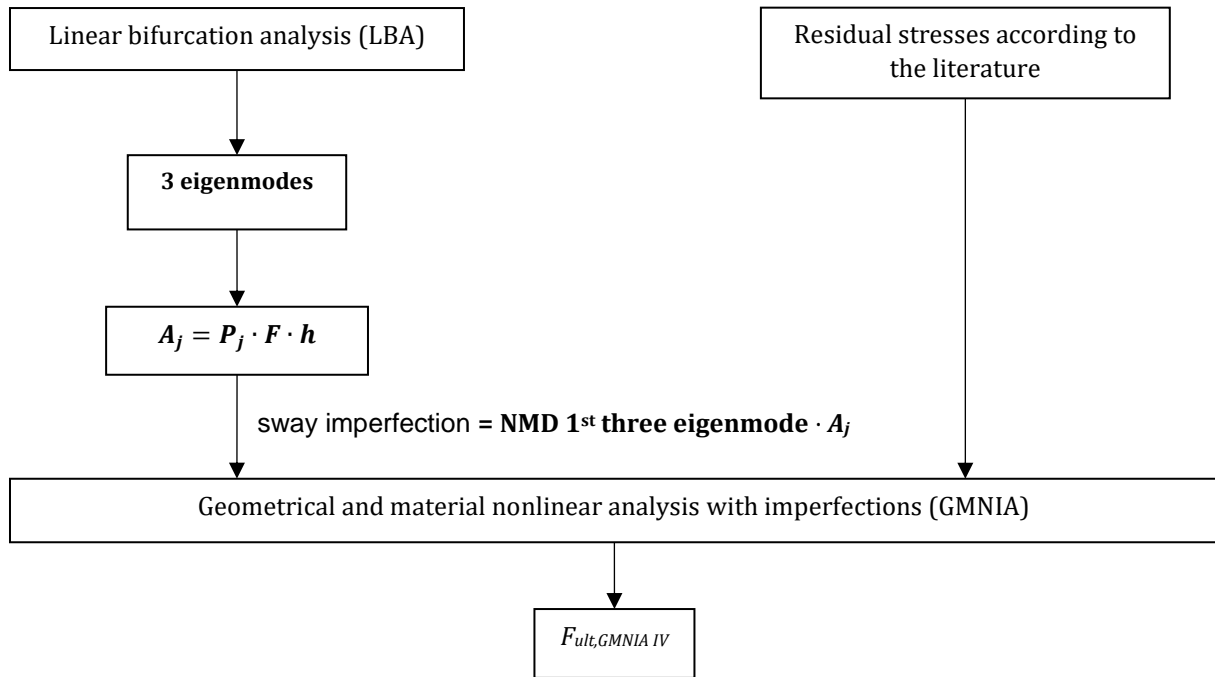


Figure 24: Flow chart of GMNIA IV for obtaining $F_{ult,GMNIA IV}$

4 FINITE ELEMENT MODEL

In this chapter, the Finite Element Model that will be used to carry out the various analyses will be discussed. The LBA and GMNIA for a frame as discussed in section 3.1 can both be performed with beam elements and shell elements. The difference between the type of elements is that beam elements only offer the ability to create a model based on the centerline of the cross-section, where a shell element can model the frame more accurately. This allows shell elements to model the flanges and web separately, which allows the implementation of residual stresses. Beam elements offer a more general approach with less detailed information regarding the frame but are still be able to apply sway and bow imperfections, which is prescribed by the EN 1993-1-1. As discussed before the sway and bow imperfections of the EN 1993-1-1 are equivalent imperfections where residual stresses are taken into account. This means that modeling residual stresses with shell elements reflects 'reality' better. According to Taris [10], modeling with shell elements has the following advantages:

- supports can be placed exactly at the desired location;
- connections between columns and beams can be modeled like real connections;
- local buckling can occur (when relevant).

In this chapter, the FE-Model with shell elements will be extensively discussed to ensure the accuracy of the model. In the end, a comparison will be made between beam and shell elements to verify the global correctness of the shell element model.

4.1 ELEMENT TYPE

For shell elements, Abaqus offers three different calculation methods that are based on two different theories. The first theory is the Kirchhoff theory, which is suitable for thin plates and shells in a small deflection situation. The Kirchhoff theory assumes that the straight lines perpendicular to the mid-surface remain linear after deformation and will stay perpendicular to the mid-surface. Therefore the elongation influence along the mid-surface deflection may be neglected [11]. The theory also makes use of the kinematic assumption that the deformation of the element does not lead to a change of thickness in the element. The second theory is the Mindlin theory, which, instead of the Kirchhoff theory, includes shear deformation and rotary inertia effects. This is achieved by the fact that the normal straight line to the mid-surface remains straight but not necessarily perpendicular to the mid-surface after deformation. This leads to including shear deformation and rotary inertia effects in the calculation. This theory is interesting for thick elements. For both theories, Abaqus offers a separate calculation method, but also offers the ability to combine the theory. This calculation method is called the general-purpose and uses thick shell theory as the shell thickness increases and uses Kirchhoff thin shell theory as the thickness decreases [12]. Abaqus assumes a shell to be thin when the thickness-to-span ratio is smaller than 1/15. For the frame dimensions considered this leads to the Kirchhoffs theory. These elements are also called S4-elements in Abaqus, where the 4 indicates the number of nodes per element.

4.2 RELEVANT ANALYSE METHODS OF ABAQUS

4.2.1 Static, general

For a LA, the static, general step of Abaqus will be used. The static, general step is a linear or nonlinear analysis, which ignores time-dependent material effects and neglects inertia effects. It makes use of the Newton Raphson method and ends when the structure is unstable. [13]

4.2.2 Static, Riks

The static Riks analysis determines the behavior of the structure after instability and continues where the Newton Raphson fails. When the load is scaled by a single scalar parameter, the Riks method can provide solutions regardless of whether the response is stable or unstable. The single scaled load is used as an unknown and instead, the arc length is used as a measure along with the result. This makes the Riks method suitable for limit load problems since the complete load-displacement path is obtained, where the ultimate load that the frame can carry be obtained from.

4.2.3 Linear perturbation buckling

In a linear perturbation buckling analysis, Abaqus determine the eigenvalue for which the model stiffness matrix becomes singular. This results in the eigenvalue where the structure reaches the Euler buckling load referred to as α_{cr} . This analysis method will be used for LBA. [14]

4.3 SHELL ELEMENT MODEL

4.3.1 Parts of the frame

The model is shown in Figure 27, which is a three-dimensional view of (all) the frames in the research-scope as discussed in section 1.4 (Figure 9), with variable conditions. There are several ways to model the complete frame, but in the end, the most realistic one should be used. To achieve this, the model is divided into different parts (columns, beams, and stiffeners), where the dimensions of these parts are based on the probable factory dimensions. The assumption is made that the columns will be manufactured till the top of the frame, where the beams will be placed between the columns. This principle is indicated by the arrows in Figure 25 and Figure 26. To achieve the most realistic joint, stiffeners will be added to the joint, which can also be seen in Figure 25 and Figure 26. The stiffener placed in the leaning column is placed to prevent local buckling in the model, in reality, this stiffener will not be there.

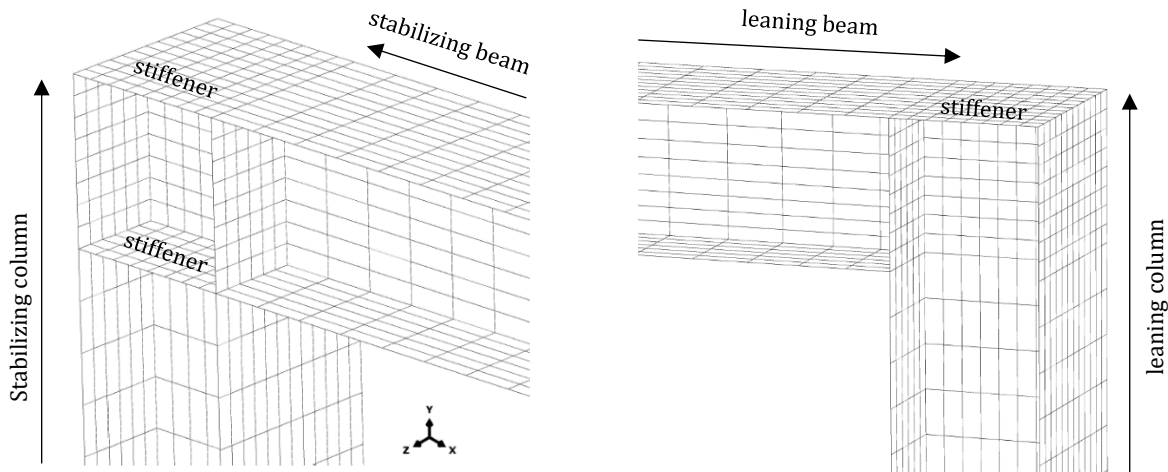


Figure 25: Column-beam joint for the stabilizing frame (left) and leaning column (right) in the FE-model

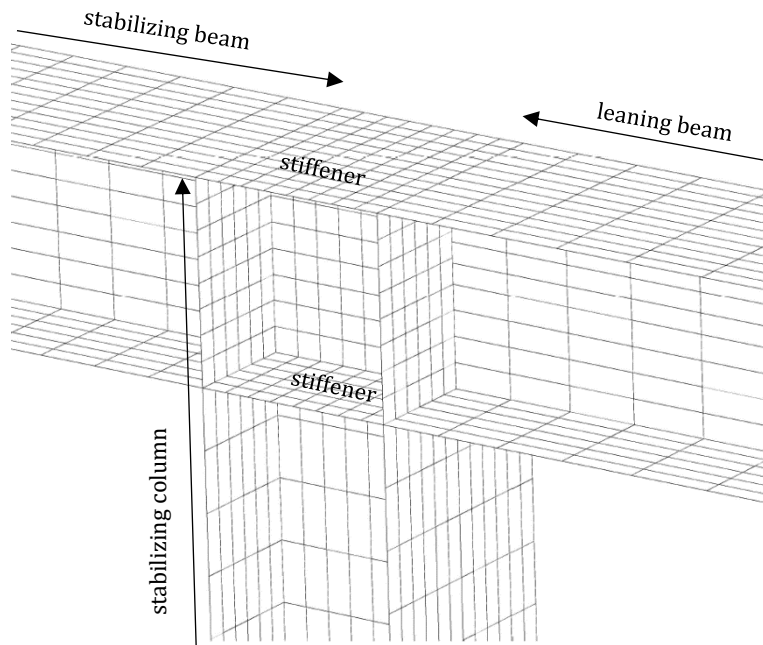


Figure 26: Beam-column-beam joint in the FE-model

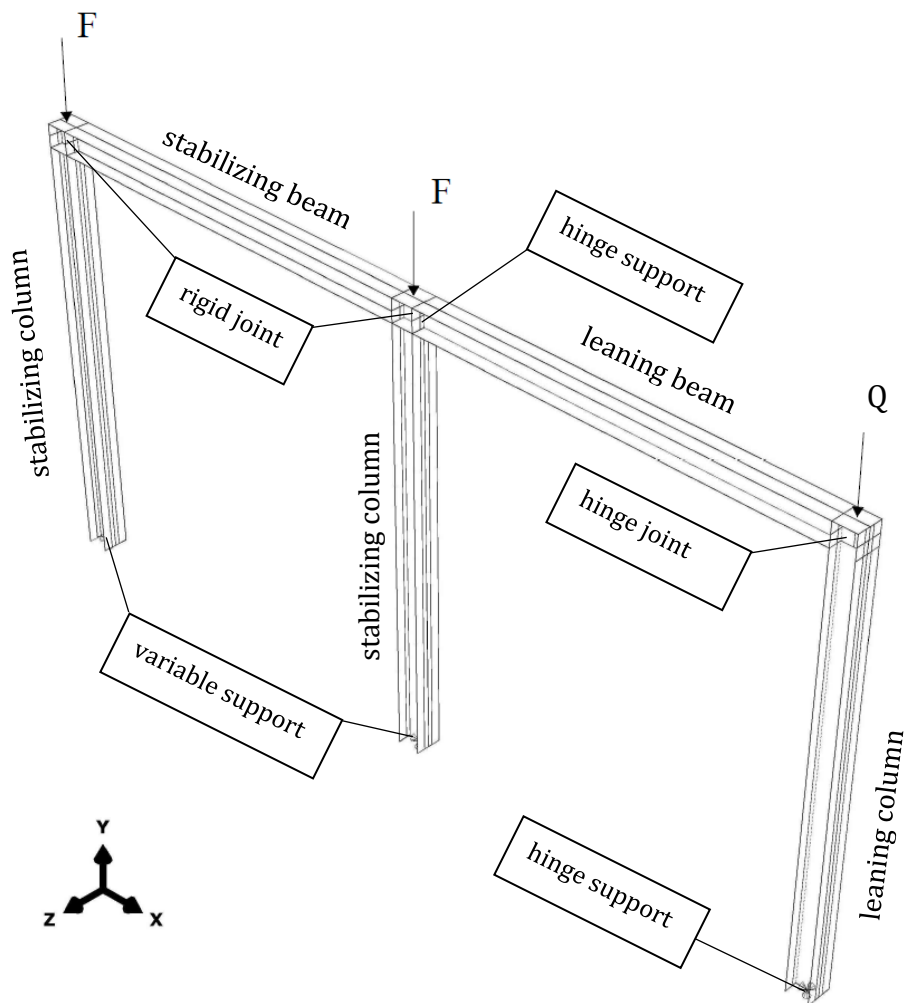


Figure 27: Variable FE-Model corresponding to the different mechanical schematizations as shown in section 1.4

4.3.2 Interaction of parts

4.3.2.1 General

The members of the frame in the FE-model, also called parts, will be assembled to create the actual model of the structure. To connect the parts and make them work together, interactions between the parts have to be implemented in the model. To simulate the correct structural behavior, different types of interaction will be used. In this research, the coupling and tie constraint is used. The coupling constraint couples the behavior and movement of a collection of nodes to one reference point. As experienced by Taris [10], supports or loads on one node of the shell element can lead to local buckling. The coupling constraint can avoid this structural behavior since it will let the cross-section behave as a whole while transferring the load to the support on one node and applying the load on one node. The option will also be used for situations where a perfectly hinged joint is desired. The coupling can act as an endplate, transferring the forces to one node. This node can be used as a perfect hinge. For the coupling type, the kinematic coupling type is used. The kinematic coupling type constrains the behavior and movement of the selected nodes to the rigid body of the reference node, where the degrees of freedom can be applied with respect to the local coordinate system of Abaqus [15].

The coupling constraint as just discussed only couples nodes of one particular part together per coupling. With the tie constraint, the surfaces and/or nodes of different parts can be tied together. Each node in the slave surface (second set of nodes or surfaces) will follow the master surface (first set of nodes or surfaces) by having the same values for its degrees of freedom to the closest master surface. By default the tie constraint accounts for shell element thickness, meaning that the actual top or bottom surface is considered while tying the closest surfaces. The rotation of the degrees of freedom from the slave surface is by default constrained to the master surface, leading to a rigid connection. When a hinge connection is desired, a constraint ratio can be specified, allowing the node to rotate.

4.3.2.2 Perfect hinge for leaning column connection

To obtain a leaning-column situation that can be compared with a two-dimensional mechanical scheme, the leaning frame should be connected to the stabilizing frame as a perfect hinge. To obtain a perfect hinge the end-nodes of the beam should be coupled to the center-node of the cross-section, as shown on the left in Figure 28. This gives the same principle as a head plate connected to the beam. The center-node should be tied to the column. This interaction between beam and column should happen at the same location as the center-node of the beam, therefore the node in the top-flange of the column should be selected. This is illustrated on the right of Figure 28, where can be seen that an area of the top-flange of the column is connected to the center-node of the beam. By default Abaqus only constrains the nodes that connect, meaning that only the node at the center-node of the beam is connected to the beam, which should be the case.

When plotting the deformations of a frame one can see that the actual deformations indicate that the joint is a perfect hinge. This can be seen in Figure 29, where the gap indicates that only the center-node is connected.

4.3.2.3 Rigid connection for stabilizing frame joint

With only a tie constraint, a rigid connection between the stabilizing- beam and column can be created. Figure 30 shows that all intersections between the column and beam (left) and stiffeners and column (right) are tied together to create a rigid connection. Note that for the column-beam intersection only the intersecting nodes are connected.

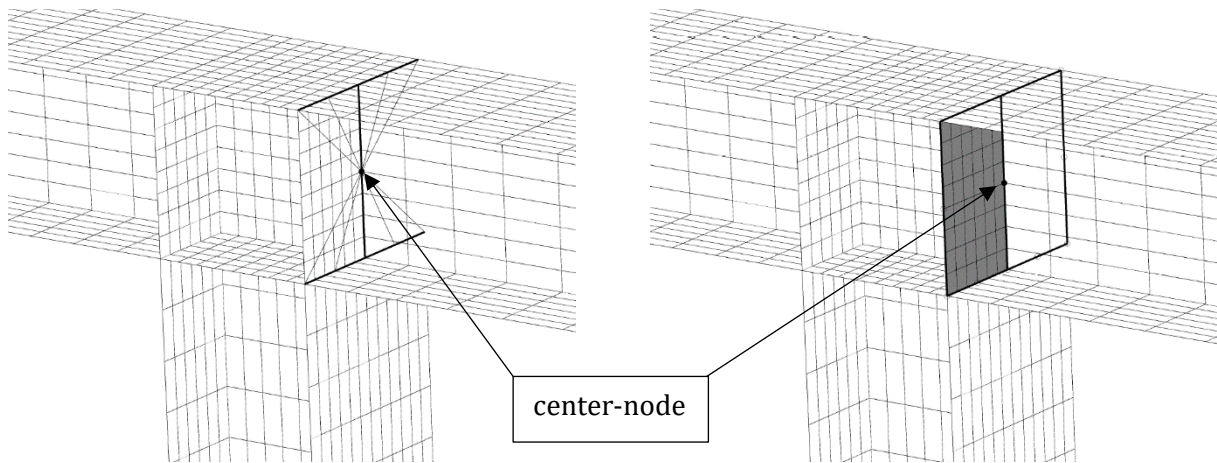


Figure 28: Coupling constraint (left) and tie constraint to one node (right) to create a hinge connection

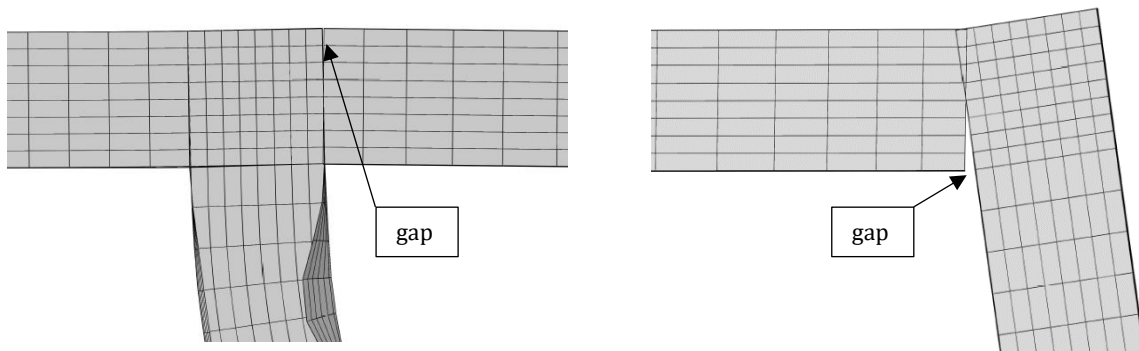


Figure 29: Deformed structure which indicates that the hinged joint act as a perfect hinge

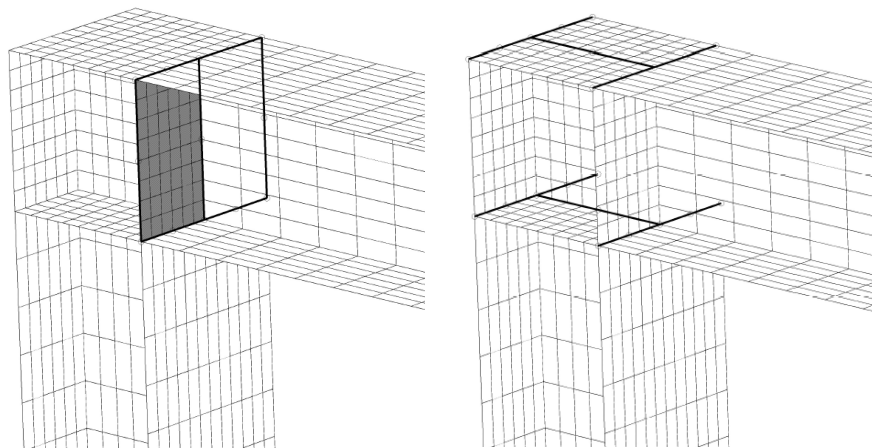


Figure 30: Tie constraints between column-beam (left) and stiffeners-column (right) to create a rigid joint

4.3.2.4 Applied conditions

Boundary conditions and load conditions should be applied to the centerline of the structure, otherwise, the model can not be compared with a two-dimensional mechanical scheme. To accomplish this, the conditions will be applied at one particular node, comparable with the perfect hinge at the leaning beam connection as just discussed above. When considering this way of modeling, measures should be taken, otherwise, the stresses that will occur at the particular node

will lead to local buckling of the web. For the top of the column, this means that the applied load will lead to local buckling in the middle of the web. To avoid this, the nodes of the web in the joint are coupled to one particular node, where the load will be applied. Due to this coupling, the load is spread over the nodes of the web in the joint. This is shown on the left of Figure 31. Additionally, this way of modeling also induces a stiff joint, corresponding to the mechanical scheme. To avoid local buckling at the bottom of the column due to the boundary condition, the nodes of the cross-section are coupled to one particular node at the center of the cross-section. This is shown on the right of Figure 31.

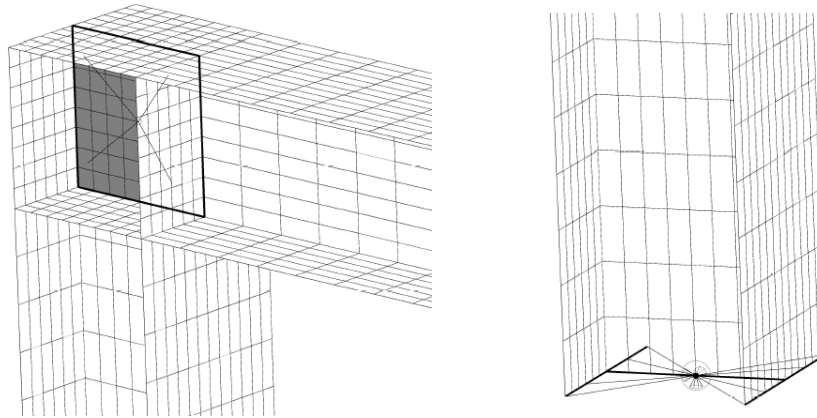


Figure 31: Coupling of nodes to apply the load conditions (left) and the boundary conditions (right)

4.3.2.5 Imperfections

Section 3.2 (Figure 14) showed that equivalent bow imperfections (GMNIA I) can be induced on the structure by an equivalent line load. Since Abaqus does not give the ability to apply line loads on shell elements, the use of coupling nodes will be used again. By coupling nodes over the height of the structure to one particular reference node, a point load equivalent to the sum of the line load can be applied at this reference node to induce the bow imperfection, as shown in Figure 32. The nodes that should be coupled are the web-flange intersection nodes at one side of the cross-section so that the load goes directly into the web. If nodes on the flanges are coupled and loaded, the flanges will deform, which is not desired.

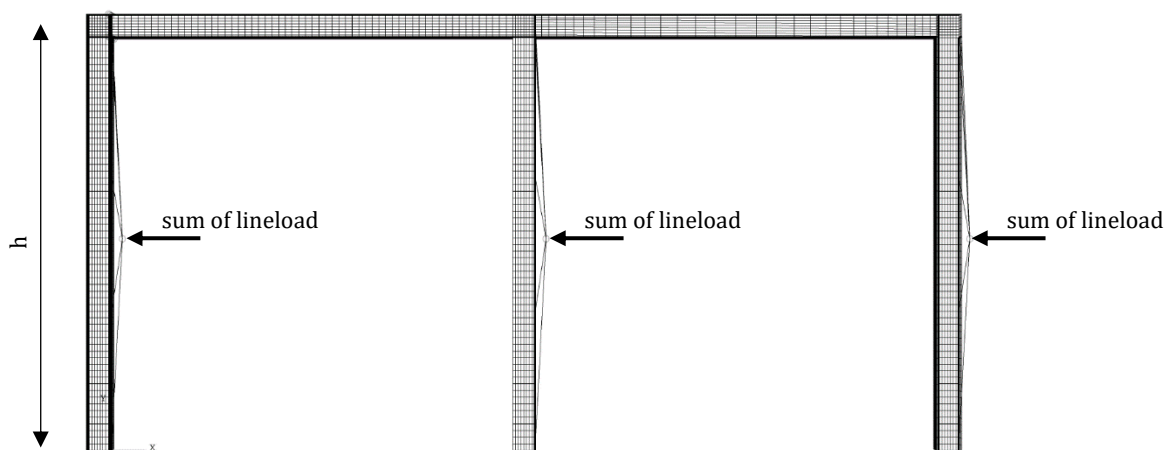


Figure 32: Bow imperfection induced by equivalent point-load

4.3.3 Supports

It is stated that out-of-plane deformations will be prevented in this research. Therefore only an in-plane GMNIA will be performed. To prevent out-of-plane deformation, all nodes of the webs for

the beam and columns need to be fully laterally supported. Only the webs must be constrained out-of-plane, so transverse strain can occur freely [10]. For hinge connected frames the simulation of the column must allow the rotation of the cross-section of the member to obtain the correct buckling mode. This can be achieved with the use of a single column point fixation at the bottom of the column, as discussed in section 4.3.2. Both boundary conditions are shown in Figure 33.

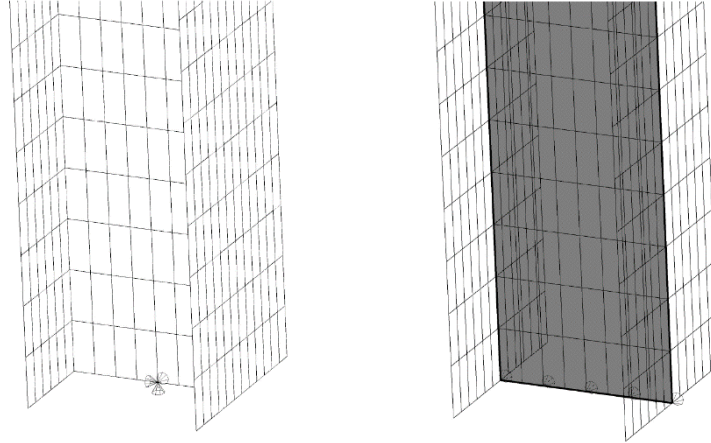


Figure 33: Boundary conditions in X and Y direction (left) and Z direction (right)

4.3.4 Cross-section

A I- or H-section in shell-elements is an assembly of two flanges and a web, all rectangular. But when we look at the hot-rolling manufacturing process of an I- and H-section, this process induces a root radius in the corners of the web on the flanges due to the shape of the press. A complication of modeling shell-elements is that this root radius cannot be modeled. This results in cross-sections that differ somewhat from reality. However, since this research is about the stability of a frame structure, and not focused on a cross-section specific case, the root radius will not be modeled. In the literature has been found that Taris [10] created an overlap between the web and the flanges in the cross-section to compromise the area of the cross-section in the FE-Model compared to the actual cross-section. However, this method has not been proven to be correct and therefore a model without overlap is used in the FE-model, meaning that it differs from the actual cross-section. Figure 34 illustrates the differences between the cross-sections when considering a HEA 300.

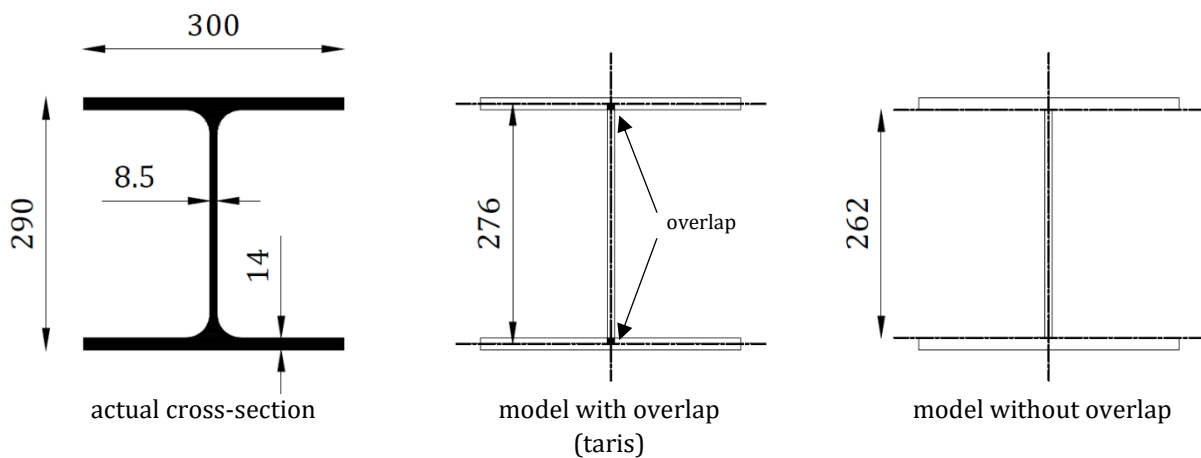


Figure 34: Actual HEA300 cross-section (left), HEA300 FE-model with overlap according to Taris (middle), and HEA300 FE-model without overlap

4.3.5 Mesh

The mesh is responsible for the accuracy of the results that are obtained from the FE-model. When a mesh has incorrect element shapes regarding the structure or the mesh is too coarse, it can lead to unreliable and unsafe results. In the validation- and case study (chapter 5 and 6) the size of the mesh will be studied to assess what the influence of the mesh-density is on the results. Based on the mesh size, the residual stress pattern is averaged over the mesh elements.

4.4 COMPARISON

The shell model that is used to carry out the case study in chapter 6 is in the process of modeling verified and compared to a beam model. This is considered to verify the outcome of the beam element model and the shell element model. For the beam elements, the Euler-Bernoulli theory is used, which results in B23 elements. From Figure 35 it can be seen that the differences between the model are minimal, with a maximum difference of the ultimate load of 2%. The analysis method for both models is a GMNIA, performed with the Static, Riks step. From Figure 35 it can be seen that the difference between behavior after instability is again minimal, where the shell elements have a larger range. Note that the results of this graph are intermediate and are only for verifying the different element models.

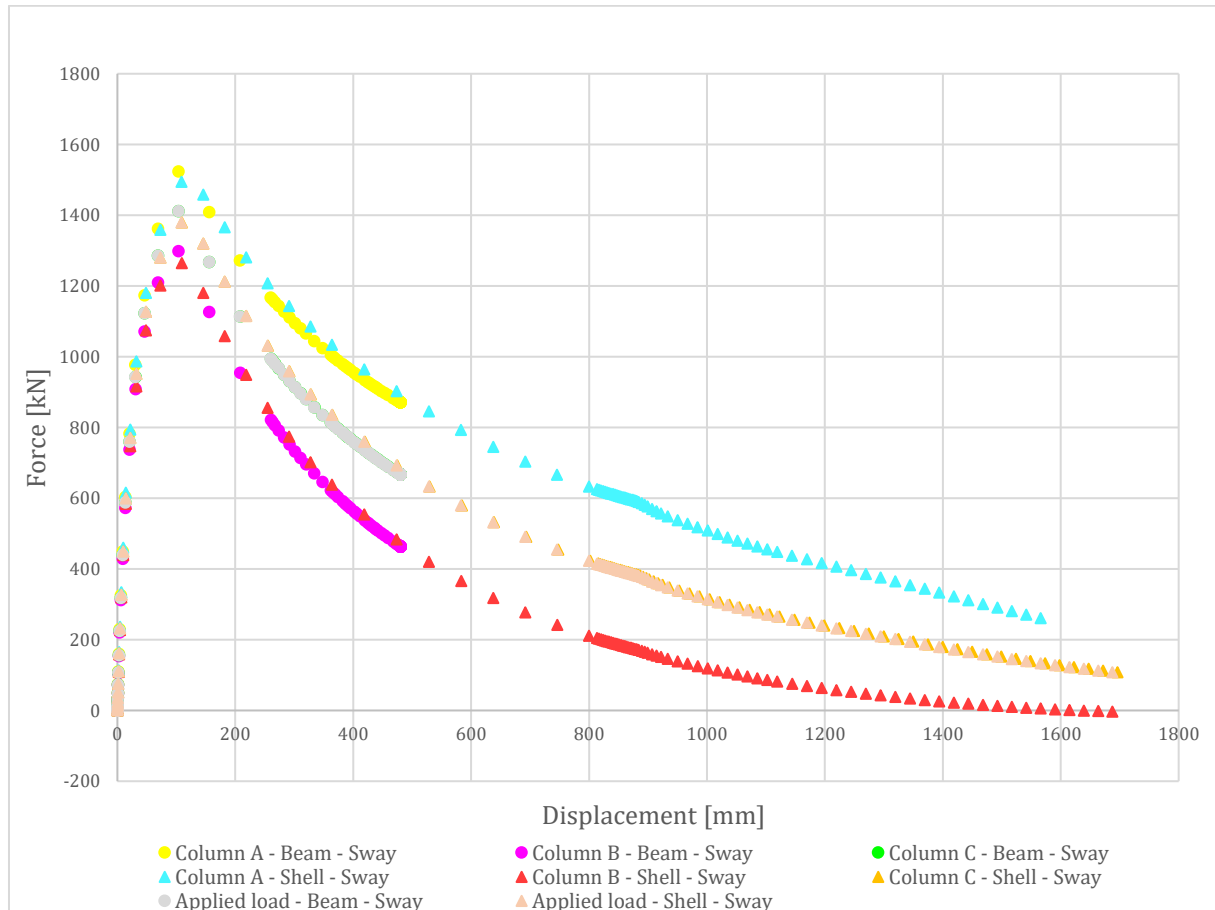


Figure 35: Comparison shell and beam elements with sway imperfections

5 VALIDATION STUDY SWAY FRAME

5.1 INTRODUCTION

In section 4.4 already a small validation is elaborated to validate the chosen elements. In this chapter, a complete validation study will be elaborated over a case from the literature. This validation case will be modeled with the methods as described in chapter 4 and analyzed with the various analysis methods as described in section 3.3. The aim is to validate the correctness and accuracy of the Finite Element Method (FEM) calculations. The literature that is used is a study from Taris [10], where an unbraced laterally supported steel sway-frame is modeled and compared to other literature, aiming to validate the scope of the general method of Eurocode 1993-1-1. The sway-frame that is used is a simple stabilizing structure consisting of two columns and a beam, where the top of the columns and the end of the beam are rigidly connected and the bottom of the column is pin-ended. At half of the column and each quarter of the beam, the frame is restricted in the Z-direction (out-of-plane). The frame is loaded with a vertical uniformly distributed load along the beam and horizontal point load in X-direction (in-plane) at the center of the left column-beam joint. The dimensions of the frame and the cross-sections are shown in Figure 36. The material is an elastic-perfectly plastic material with a yield-strength of 275 N/mm², considered with an engineering stress-strain diagram.

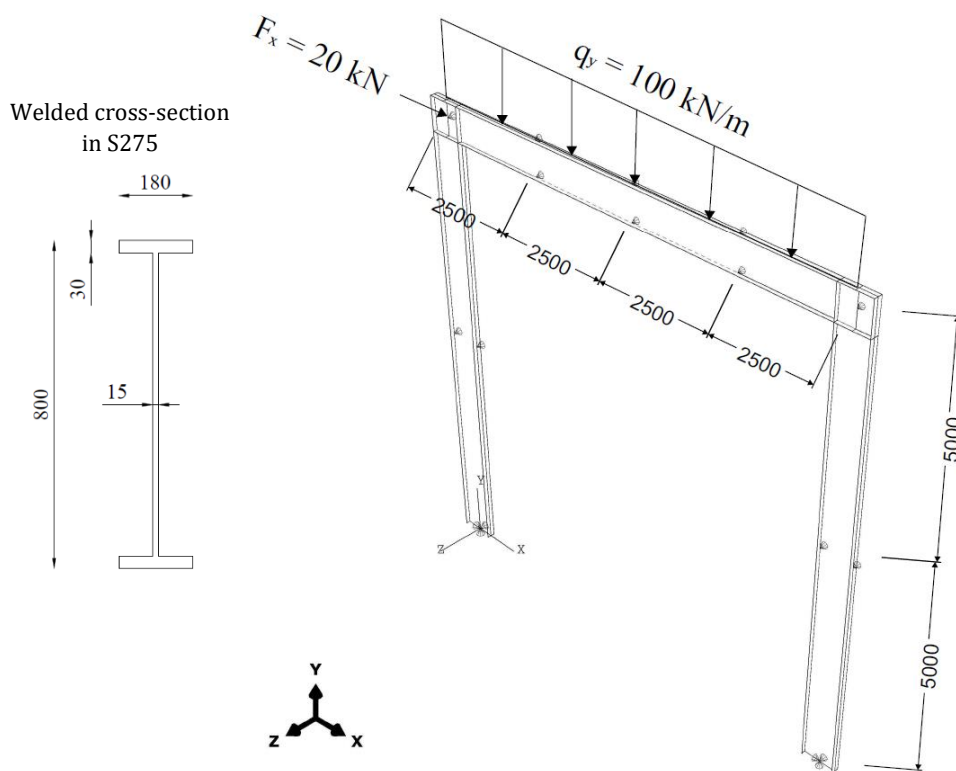


Figure 36: Three-dimensional illustration of the dimensions of the considered frame

5.2 ANALYSIS METHODS AND IMPERFECTIONS

The frame in this validation study is not fully restricted in the out-of-plane direction, but only at various nodes. This allows deformation and buckling in the weak axis of the frame and therefore also out-of-plane imperfections are taken into account in this chapter. The magnitudes of the imperfections will be as described in section 3.3. However, to make the results of the FEM calculations comparable with the work of Taris [10], the imperfections for the various analyses

are also based on the imperfections of Taris. Taris only used sway-imperfections in his study and therefore GMNIA I is only performed with an equivalent sway imperfection. Analysis method GMNIA II is performed three times, where the imperfections for the first one (GMNIA II) is based on section 3.3.2, and for the other two analyses (GMNIA IIa and GMNIA IIb) the imperfections are derived as Taris [10]. Table 2 shows the imperfections and their magnitudes that are used in the validation study. Analysis method GMNIA IV is not used by Taris and makes it that GMNIA II and GMNIA IV give unique results and are not directly comparable with Taris.

Table 2: Imperfections that are used in the various analyses in the validation study

ANALYSES	IMPERFECTIONS		RESIDUAL STRESSES
	In-plane	Out-of-plane	
LBA	-	-	-
GMNIA I	In-plane sway imperfection according to EN 1993-1-1 Clause 5.3.2(3) induced with equivalent forces	-	NO
GMNIA II	In-plane sway imperfection according to EN 1993-1-1 Clause 5.3.2(3) induced with equivalent forces	First out-of-plane eigenmode sized according to EN 1993-1-1 Clause 5.3.2(11) (η_{init}) induced with the normalized matrix of displacements	NO
GMNIA IIa	In-plane sway imperfection according to EN 1993-1-1 Clause 5.3.2(3) induced with equivalent forces	First out-of-plane eigenmode sized according to EN 1993-1-1 Clause 5.3.2(3) induced with the normalized matrix of displacements	NO
GMNIA IIb	In-plane sway imperfection according to EN 1993-1-1 Clause 5.3.2(3) induced with equivalent forces	First out-of-plane eigenmode sized as sway imperfection according to NEN 6771 induced with the normalized matrix of displacements	NO
GMNIA III	In-plane sway imperfection sized as $L_{cr}/1000$ induced with equivalent forces	First out-of-plane eigenmode sized as $L_{cr}/1000$ induced with the normalized matrix of displacements -	YES
GMNIA IV	First in-plane eigenmode sized as $A = P \cdot F \cdot h$ induced with the normalized matrix of displacements -	First three out-of-plane eigenmodes sized as $A_j = P_j \cdot F \cdot h$ induced with the normalized matrix of displacements -	YES

5.3 MESH REFINEMENT STUDY AND THE CORRESPONDING RESIDUAL STRESSES

Taris [10] made use of shell281 elements in Ansys. These elements are an eight-node element with six degrees of freedom at each node and are suitable for thin to moderately-thick shell elements. As described in section 4.1, the elements that are used for this validation study are a four-node S4 shell-element in Abaqus. To ensure the accuracy of the FE-model, the mesh-size that is needed for the four-node S4 elements is studied with a mesh-refinement study. The mesh-refinement study is performed for an elastic problem and a plastic problem. The reason for that is that since the considered frame is an instability problem, the deformation is hard to compare. The elastic problem is studied with an LBA, where the critical force amplifier α_{cr} is compared for the different mesh-sizes. For the plastic problem, an in-plane GMNIA (GMNIA I) is performed and the reaction force (RF) in kN and the load proportionality factor (LPF) at the ultimate load are compared. In Table 3 and Table 4, the results of the mesh-refinement study are shown. In both

studies, the mesh size of Taris is taken as a starting point (most coarse mesh). Both mesh-refinement studies result in a reliable mesh-size of $48.125 \times 48.125 \times 45$ (axbxc), which is the mesh-size that will be used for this validation study. This mesh size is labeled as a coarse mesh and will be used for all analyses, except GMNIA IIIb. For GMNIA IIIb the finest mesh will be used. The mesh size direction of the coding a, b and c are shown in Figure 37.

Table 3: Results for the elastic mesh-refinement study for the validation study

axbxc	Number of elements	Rel. mesh density $\frac{x}{385 \times 192.5 \times 45}$	Critical force amplifier (α_{cr})	CPU time	% Error $\frac{24.063 \times 24.063 \times 22.5 - X}{24.063 \times 24.063 \times 22.5}$	
	[-]	[-]	[-]	[s]	[%]	
24.063x24.063x22.5	65468	61.1	2.6004	520.1	0.00%	
48.125x48.125x45	16240	15.15	2.6057	118.6	-0.2%	
192.5x96.25x45	2832	2.64	2.6798	14.2	-3.05%	
385x192.5x45	1072	1	2.8163	6.2	-8.30%	

Table 4: Results for the plastic mesh-refinement study for the validation study

axbxc	Rel. mesh density $\frac{x}{385 \times 192.5 \times 45}$	Reaction force (RF)	Load proportionality factor (LPF)	CPU time	% Error $\frac{6.25 \times 6.25 \times 6.25 - X}{6.25 \times 6.25 \times 6.25}$	
	[-]	[kN]	[-]	[s]	RF (%)	LPF (%)
24.063x24.063x22.5	61.1	1145.1	2.5100	4689.2	0.00%	0.00%
48.125x48.125x45	15.15	1148.7	2.5192	1032.5	-0.32%	-0.37%
192.5x96.25x45	2.64	1160.2	2.5415	171.7	-1.32%	-1.26%
385x192.5x45	1	1173.0	2.5665	76.8	-2.44%	-2.25%

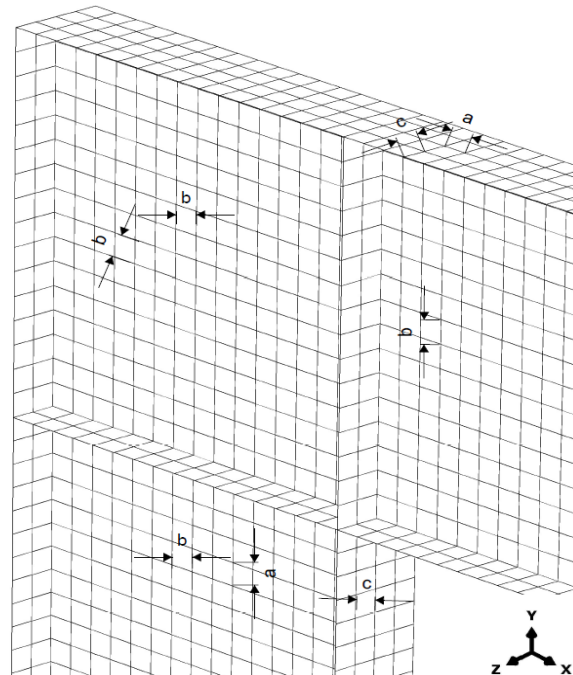


Figure 37: Mesh-size coding illustrated on the left column-beam joint

Analysis methods GMNIA III (a and b) and GMNIA IV include residual stresses that will be applied to each mesh element in FEM. The pattern of the residual stresses is dependent on the mesh size. Figure 38 shows the residual stress patterns for the actual profile, the coarse mesh, and the fine mesh for welded cross-sections. According to Table 3 and Table 4, the eight-node S8 elements that Taris used, allowed him to use a more coarse mesh than the S4 element used in this research. For comparison reasons, the mesh that will be used for the S4 elements will follow the same coarse mesh pattern as Taris [10], as shown in Figure 38. To see the influence of the results, also the finest mesh pattern is used (labeled as fine mesh) to come closer to the actual residual stress pattern.

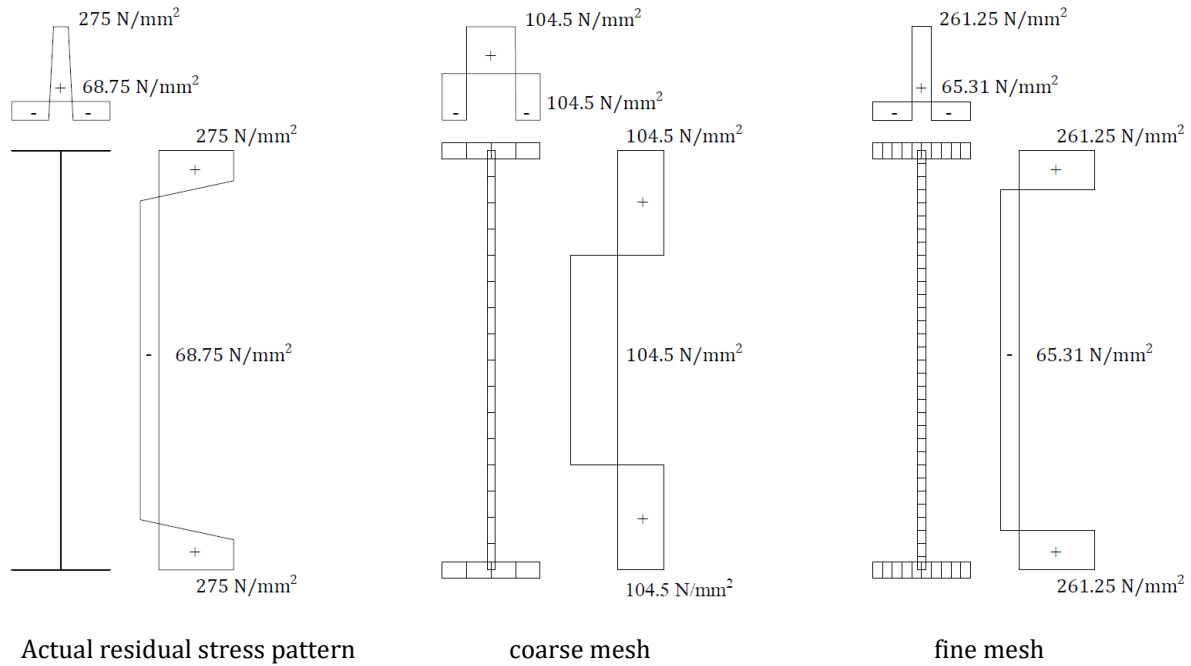


Figure 38: Residual stress pattern for the actual profile (left), coarse mesh (middle), and fine mesh (right)

5.4 NUMERICAL RESULTS OF THE VARIOUS ANALYSES - VALIDATION STUDY

5.4.1 LBA

The LBA is performed for different purposes. The first purpose is to obtain α_{cr} , which can be compared to other LPF but also be used to determine L_{cr} . The other purpose is to determine the input for the other various analyses. The first input is the normalized matrix of displacements for the governing critical eigenmode (in-plane and out-of-plane) and the second input is the bending moment at the critical cross-section due to the governing critical eigenmode ($EI|\eta_{cr}''|_{max}$ in GMNIA II).

The critical eigenmode results in a load factor of $\alpha_{cr,z} = 2.61$, as shown on the left in Figure 39. It appears that the first eigenmode is also the critical eigenmode for out-of-plane buckling, which results in the normalized matrix of displacements as shown on the left of Figure 39. The first in-plane buckling mode is found at eigenmode 23 with a load factor of $\alpha_{cr,x} = 14.56$, shown on the right of the figure. The corresponding $EI|\eta_{cr}''|_{max}$ at the critical cross-section is $EI|\eta_{cr}''|_{max} = 1.382 \cdot 10^6$, as shown in Figure 40.

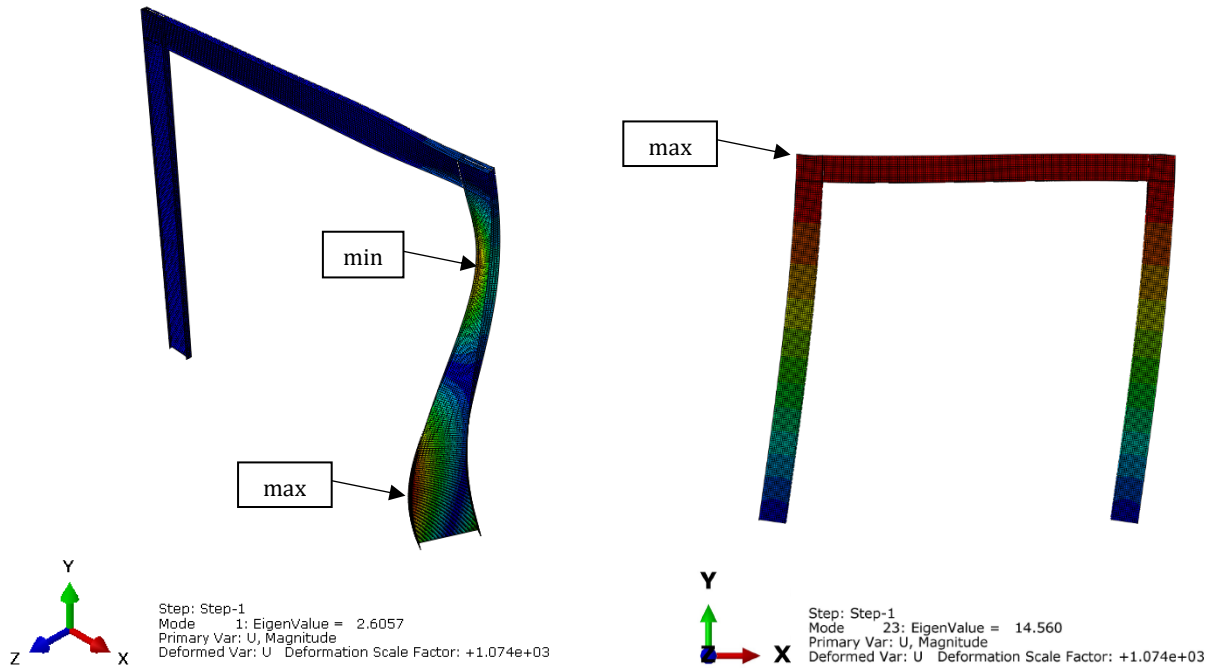


Figure 39: First out-of-plane eigenmode (left) and in-plane eigenmode (right)

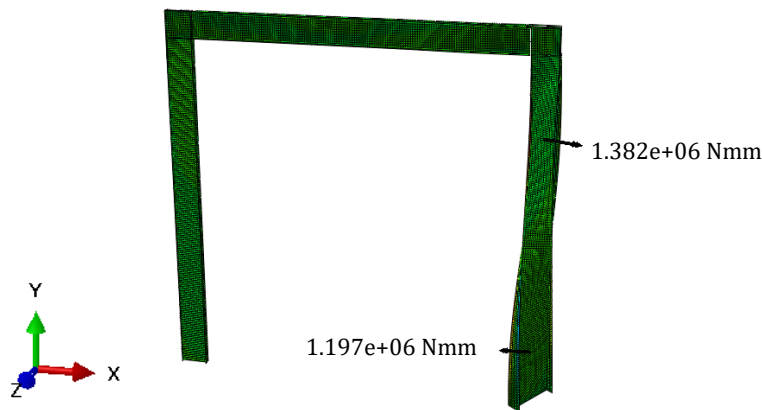


Figure 40: Resultant of section-moment $EI|\eta''_{cr}|_{max}$ plotted on the deformed state

5.4.2 GMNIA I according to EN 1993-1-1 clause 5.3.2(3) (in-plane)

For GMNIA I, only a sway imperfection according to EN 1993-1-1 clause 5.3.2(3) is used, and is given as the following:

$$\begin{aligned} \phi_0 &= 1/200 \\ \alpha_h &= \frac{2}{\sqrt{h}} = \frac{2}{\sqrt{10}} = 0.632, \text{ where } 0.632 \leq \frac{2}{3} \rightarrow = \frac{2}{3} \\ \alpha_m &= \sqrt{0.5 \left(1 + \frac{1}{m}\right)} = \sqrt{0.5 \left(1 + \frac{1}{2}\right)} = 0.886 \\ \phi &= \phi_0 \cdot \alpha_h \cdot \alpha_m = \frac{1}{200} \cdot \frac{2}{3} \cdot 0.886 = 2.89 \cdot 10^{-3} \\ \Delta_x &= 2.89 \cdot 10^{-3} \cdot 10000 = 28.9 \text{ mm} \end{aligned}$$

The in-plane sway imperfection Δ_x is induced on the structure as an equivalent force as shown in Figure 14. This in-plane GMNIA results in an LPF of $\alpha_{GMNIA I} = 2.52$. The load-displacement path

corresponding to this LPF is shown in Figure 41, where each dot is a converged equilibrium state. The displacement taken is the lateral displacement in the X-direction at the left top node. The result of the LBA is also shown in Figure 41.

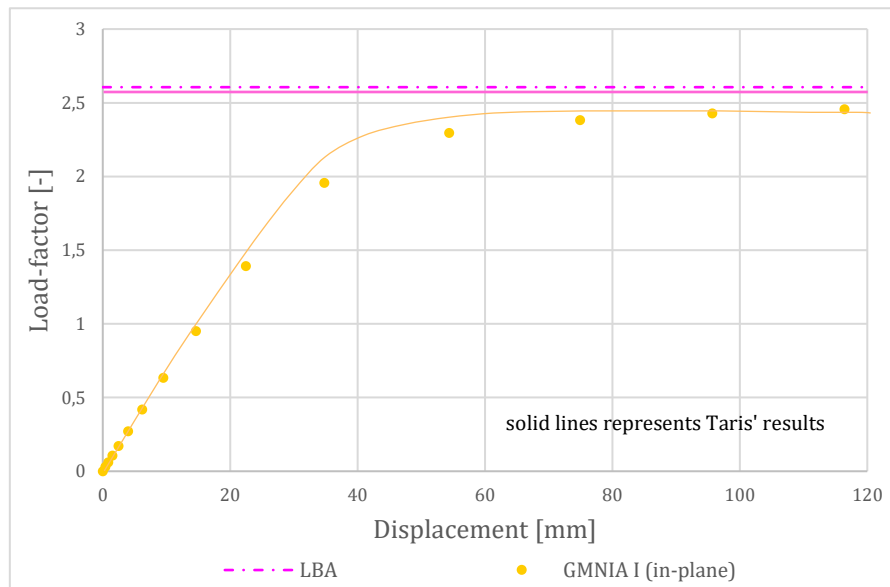


Figure 41: Load displacement diagram for GMNIA I, including the buckling load from the LBA

5.4.3 GMNIA II according to EN 1993-1-1 clause 5.3.2(11)

For GMNIA II, the same in-plane imperfection is taken as GMNIA I, namely the equivalent sway imperfection of the EN 1993-1-1 clause 5.3.2(3). This results in:

$$\Delta_x = 28.9 \text{ mm}$$

For the out-of-plane imperfection, the normalized matrix of displacements of the first out-of-plane eigenmode as shown in Figure 39 is used, sized according to the EN 1993-1-1 clause 5.3.2(11). In Taris [10] can be found that for the section modulus the plastic section modulus $W_{z,pl}$ is used. However, this does not match with the cross-section class to the corresponding geometry of the cross-section. Since this chapter is only a validation study about the FE-Model and not the actual calculation, $W_{z,pl}$ is also used in this study. The scale-factor of the imperfection is determined by the following:

$$W_{z,pl} = 2 \cdot (740 \cdot 7.5 \cdot 3.75 + 2 \cdot (90 \cdot 30 \cdot 45)) = 527625 \text{ mm}^3$$

$$M_{Rk,z} = f_{y,d} \cdot W_{z,pl} = 275 \cdot 527625 = 145.1 \cdot 10^6 \text{ Nmm}$$

$$N_{Rk} = f_{y,d} \cdot A = 275 \cdot 21900 = 6022.5 \cdot 10^3 \text{ N}$$

$$F_{cr,z} = 520000 \cdot 2.61 = 1355 \cdot 10^3 \text{ N (LBA)}$$

$$\bar{\lambda} = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr,z}}} = \sqrt{\frac{N_{Rk}}{F_{cr,z}}} = \sqrt{\frac{6022.5 \cdot 10^3}{1355 \cdot 10^3}} = 2.11 \text{ [-]}$$

$\alpha = 0.49$, according to table 1 in appendix A

$$\phi = 0.5[1 + \alpha(\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 0.5[1 + 0.49 \cdot (2.11 - 0.2) + 2.11^2] = 3.19 \text{ [-]}$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = \frac{1}{3.19 + \sqrt{3.19^2 - 2.11^2}} = 0.18 [-]$$

$$e_0 = \alpha(\bar{\lambda} - 0.2) \frac{M_{Rk,z}}{N_{Rk}} \cdot \frac{1 - \chi \bar{\lambda}^2}{1 - \chi^2 \bar{\lambda}^2} = 0.49 \cdot (2.11 - 0.2) \frac{145.1 \cdot 10^6}{6022.5 \cdot 10^3} \cdot \frac{1 - \frac{0.18^2 \cdot 2.11^2}{1}}{1 - 0.18^2 \cdot 2.11^2} = 22.53 \text{ mm}$$

$$EI |\eta_{cr}''|_{max} = 1.382 \cdot 10^6 \text{ Nmm}$$

$$\eta_{init} = e_0 \cdot \frac{F_{cr,z}}{EI |\eta_{cr}''|_{max}} \eta_{cr} = 22.53 \cdot \frac{1355 \cdot 10^3}{1.382 \cdot 10^6} = 22.09 \text{ mm}$$

The in-plane sway imperfection Δ_x is induced on the structure as an equivalent force. Value η_{init} is the magnitude to scale the normalized matrix of displacements. This GMNIA results in an LPF of $\alpha_{GMNIA II} = 1.65$. GMNIA II is the Analysis method where the normalized matrix of displacements is sized by a value determined according to EN 1993-1-1 clause 5.3.2(11). In the study of Taris, the analysis method as GMNIA II is used, but not the specific value determined according to EN 1993-1-1 clause 5.3.2(11). For that reason, the result of section 5.4.3 can not be compared directly to the result of Taris. In the upcoming sections, 5.4.4 and 5.4.5 the same determination is used but now with magnitudes as Taris, which makes them comparable to the results of Taris.

5.4.4 GMNIA IIa according to EN 1993-1-1 clause 5.3.2(3)

For GMNIA IIa, the same in-plane imperfection is taken as GMNIA I, namely the equivalent sway imperfection of the EN 1993-1-1 clause 5.3.2(3). This results in:

$$\Delta_x = 28.9 \text{ mm}$$

The critical out-of-plane eigenmode that is obtained from the LBA (left of Figure 39) is somewhat shaped like a bow imperfection. According to EN 1993-1-1 clause 5.3.2(3), the bow imperfection for cross-section with buckling curve C can be shaped as $L_{cr}/150$, which gives the following:

$$L_{cr,z} = \pi \cdot \sqrt{\frac{E \cdot I_z}{\alpha_{cr,z} \cdot N_{Ed}}} = \pi \cdot \sqrt{\frac{210000 \cdot 293.68 \cdot 10^5}{2.6057 \cdot 520000}} = 6702 \text{ mm}$$

$$e_{0,z} = \frac{L_{cr,z}}{150} = \frac{6702}{150} = 44.68 \text{ mm}$$

The in-plane sway imperfection Δ_x is induced on the structure as an equivalent force. Value $e_{0,z}$ is the magnitude factor to scale the normalized matrix of displacements of the first out-of-plane eigenmode. This GMNIA results in an LPF of $\alpha_{GMNIA IIa} = 1.48$

5.4.5 GMNIA IIb according to NEN 6771

For GMNIA IIb, the same in-plane imperfection is taken as GMNIA I, namely the equivalent sway imperfection of the EN 1993-1-1 clause 5.3.2(3). This results in:

$$\Delta_x = 28.9 \text{ mm}$$

As discussed in section 5.4.4, the critical out-of-plane eigenmode that is obtained from the LBA is somewhat shaped like a bow imperfection. According to NEN 6771, the bow imperfection can be shaped with the magnitude of the following equation:

$$e_{0,z} = \alpha_k \cdot (\lambda_{z,rel} - \lambda_0) \cdot \frac{M_{z,u,d}}{N_{c,u,d}}$$

$$L_{cr,z} = \pi \cdot \sqrt{\frac{E \cdot I_z}{\alpha_{cr,z} \cdot N_{Ed}}} = \pi \cdot \sqrt{\frac{210000 \cdot 293.68 \cdot 10^5}{2.6057 \cdot 520000}} = 6702 \text{ mm}$$

$$I_z = \frac{1}{12} \cdot 740 \cdot 15^3 + 2 \cdot \left(\frac{1}{12} \cdot 30 \cdot 180^3 \right) = 293.68 \cdot 10^5 \text{ mm}^4$$

$$i_z = \sqrt{\frac{I_z}{A}} = \sqrt{\frac{293.68 \cdot 10^5}{21900}} = 36.62 \text{ mm}$$

$$\lambda_e = \pi \cdot \sqrt{\frac{E}{f_{y,d}}} = \pi \cdot \sqrt{\frac{210000}{275}} = 86.81$$

$$\lambda_z = \frac{L_{cr,z}}{i_z} = \frac{6702}{36.62} = 183.03$$

$$\lambda_{z,rel} = \frac{\lambda_z}{\lambda_e} = \frac{183.03}{86.81} = 2.11$$

$$W_{z,pl} = 2 \cdot (740 \cdot 7.5 \cdot 3.75 + 2 \cdot (90 \cdot 30 \cdot 45)) = 527625 \text{ mm}^3$$

$$M_{z,u,d} = f_{y,d} \cdot W_{z,pl} = 275 \cdot 527625 = 145.1 \cdot 10^6 \text{ Nmm}$$

$$N_{c,u,d} = f_{y,d} \cdot A = 275 \cdot 21900 = 6022.5 \cdot 10^3 \text{ N}$$

$\alpha_k = 0.49$, according to table 25 of the NEN 6770

$\lambda_0 = 0.20$, according to table 25 of the NEN 6770

$$e_{0,z} = 0.49 \cdot (2.11 - 0.2) \cdot \frac{145.1 \cdot 10^6}{6022.5 \cdot 10^3} = 22.53 \text{ mm}$$

The in-plane sway imperfection Δ_x is induced on the structure as an equivalent force. Value $e_{0,z}$ is the magnitude to scale the normalized matrix of displacements of the first out-of-plane eigenmode. This GMNIA results in an LPF of $\alpha_{GMNIA \text{ I Ib}} = 1.64$. In the graph in Figure 42, it can be seen that the use of the Ayrton-Perry format (equation (46)) in GMNIA II and GMNIA I Ib results in an evenly ultimate LPF. From this figure, it also can be seen that the use of $L_{cr,z}/150$ results in a lower ultimate LPF, somewhat more conservative.

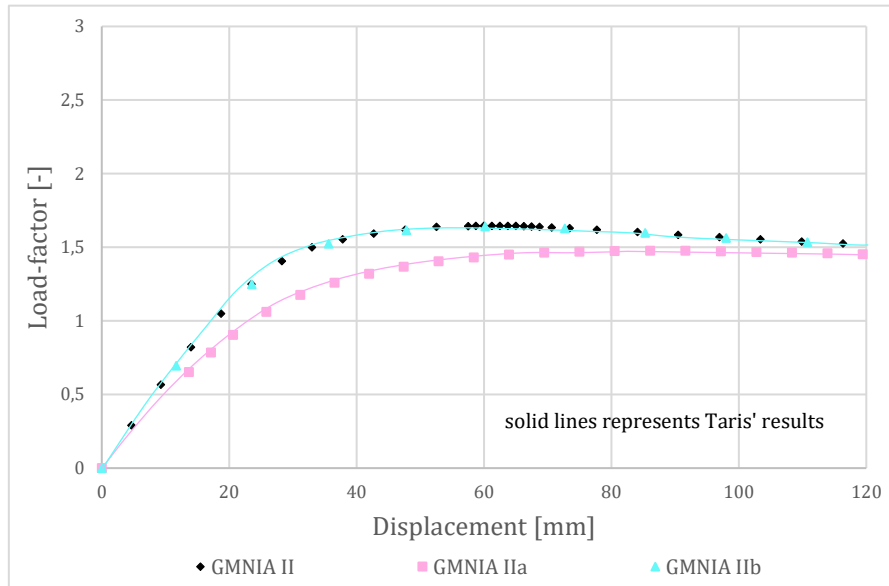


Figure 42: Load displacement diagram for GMNIA II, GMNIA IIa, and GMNIA IIb

5.4.6 GMNIA III according to Vogel et al.

For GMNIA III, the magnitude of the imperfections is derived according to Vogel et al. [8], as described in section 3.3.3. The residual stress pattern that is plotted on the structure is shown in section 5.3. The determination of the magnitude of the in- and out-of-plane imperfection to scale the normalized matrix of displacements is the following:

In-plane:

$$I_y = \frac{1}{12} \cdot 15 \cdot 740^3 + 2 \cdot \left(\frac{1}{12} \cdot 180 \cdot 30^3 + 30 \cdot 180 \cdot 385^2 \right) = 21081.7 \cdot 10^5 \text{ mm}^4$$

$$L_{cr,x} = \pi \cdot \sqrt{\frac{E \cdot I_y}{\alpha_{cr,x} \cdot N_{Ed}}} = \pi \cdot \sqrt{\frac{210000 \cdot 2108170000}{14.560 \cdot 520000}} = 24023 \text{ mm}$$

$$\frac{L_{cr,x}}{1000} = \frac{24023}{1000} = 24.02 \text{ mm}$$

The formula $L_{cr,x}/1000$ gives the sway imperfection at the middle of the buckling length. In the case of the in-plane imperfection, the middle of the buckling length does not match with the height of the frame. To obtain the in-plane imperfection at the height of the frame, a geometrical correction has to be used, as described in section 3.3.2. The geometrical correction can be derived by the following:

$$P = r - 24.02$$

$$r^2 = P^2 + \left(\frac{24023}{2} \right)^2$$

$$(P + 24.02)^2 = P^2 + \left(\frac{24023}{2} \right)^2$$

$$P^2 + 48.04P + 577.11 = P^2 + \left(\frac{24023}{2} \right)^2$$

$$P = 3002882 \text{ mm}$$

$$r = 2002906 \text{ mm}$$

$$x = \frac{24023}{2} - 10000 = 2011.6 \text{ mm}$$

$$y = \sqrt{2002906^2 - 2011.6^2} = 3002905$$

$$\Delta_x = 3002905 - 3002882 = 23.35 \text{ mm (without rounding off)}$$

Value Δ_x is the magnitude to scale the normalized matrix of displacements of the first in-plane eigenmode.

Out-of-plane:

$$e_{0,z} = \frac{L_{cr,z}}{1000} = \frac{6702}{1000} = 6.70 \text{ mm}$$

Value $e_{0,z}$ is the magnitude to scale the normalized matrix of displacements of the first out-of-plane eigenmode. The value does not have to be corrected since it is at the middle of the buckling length.

This GMNIA results in an LPF of $\alpha_{GMNIA IIIa} = 1.82$ (coarse) for the coarse mesh and $\alpha_{GMNIA IIIb} = 1.99$ (fine) for the fine mesh. For both analyses, the load-displacement path is plotted in Figure 43. It can be seen that the analysis with a finer mesh leads to a higher ultimate LPF, which is unlogic and can not be described. This will be further discussed in section 5.5.

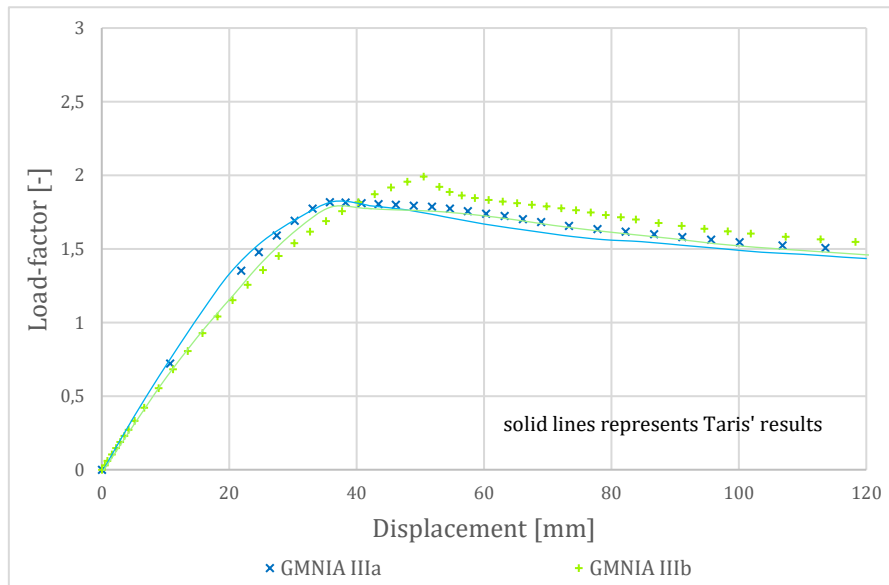


Figure 43: Load displacement diagram for GMNIA IIIa and GMNIA IIIb

5.4.7 GMNIA IV according to Shayan et al. and Vogel et al.

Analysis method GMNIA IV makes use of imperfections based on various eigenmodes with scale factors according to Shayan et al. [9], and residual stresses according to Vogel et al. [8], as described in section 3.3.4. For the in-plane imperfection, only the first eigenmode is used, since this eigenmode was the only in-plane buckling shape obtained. This eigenmode shape is the top illustration in Figure 44. The three eigenmodes that are taken for the out-of-plane direction are the first three out-of-plane eigenmodes, namely mode 1, mode 3, and mode 8 (bottom illustrations in Figure 44).

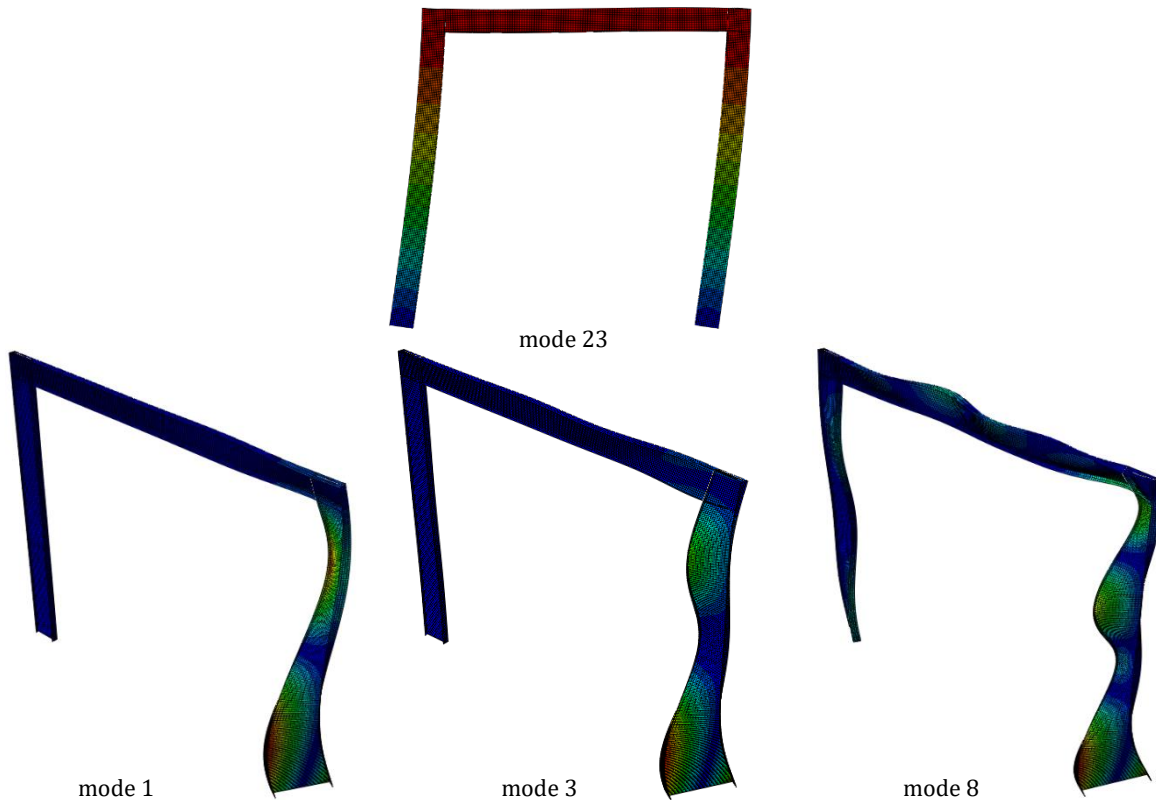


Figure 44: The first in-plane eigenmode (top) and the first three out-of-plane eigenmodes (bottom) where the normalized matrix of displacements will be scaled as imperfection in GMNIA IV

The magnitude of the in- and out-of-plane normalized matrix of displacements are based on Table 1 and given as the following:

In-plane:

$$P_{1,x} = 1 \text{ according to Table 1}$$

$$F_{shayan} = 0.001228 \text{ according to Table 1}$$

$$h = 10000 \text{ mm}$$

$$A_{1,x} = P_{1,x} \cdot F_{shayan} \cdot h = 1 \cdot 0.001228 \cdot 10000 = 12.28$$

Out-of-plane:

$$P_{1,z} = 0,623 \text{ according to Table 1}$$

$$P_{2,z} = 0,161 \text{ according to Table 1}$$

$$P_{3,z} = 0.216 \text{ according to Table 1}$$

$$F_{shayan} = 0.001838 \text{ according to Table 1}$$

$$h = 10000 \text{ mm}$$

$$A_{1,z} = P_{1,z} \cdot F_{shayan} \cdot h = 0.623 \cdot 0.001838 \cdot 10000 = 11.4$$

$$A_{2,z} = P_{2,z} \cdot F_{shayan} \cdot h = 0.161 \cdot 0.001838 \cdot 10000 = 2.96$$

$$A_{3,z} = P_{3,z} \cdot F_{shayan} \cdot h = 0.216 \cdot 0.001838 \cdot 10000 = 3.97$$

Value $A_{1,x}$ is the magnitude to scale the normalized matrix of displacements of the first in-plane eigenmode, mode 23. Values $A_{1,z}$, $A_{2,z}$, and $A_{3,z}$ are the magnitude to scale the normalized matrix of displacements of the three first out-of-plane eigenmode, modes 1, 3, and 8 respectively. This GMNIA results in an LPF of $\alpha_{GMNIA IV}=1.73$. The load-displacement path is plotted in Figure 45.

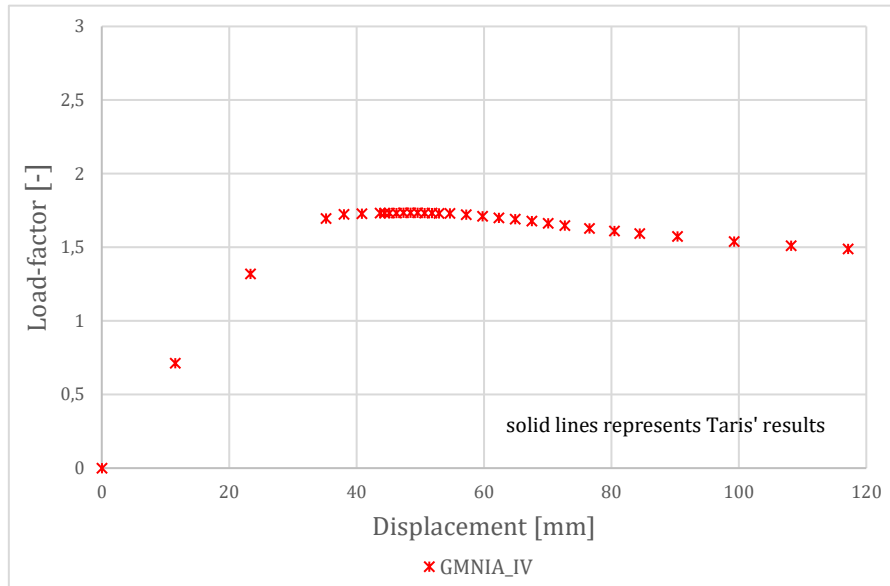


Figure 45: Load displacement diagram for GMNIA IV

5.5 COMPARISON OF RESULTS

In this paragraph, the results of the validation study and the results of Taris will be compared. Note that this paragraph is only to validate the FEM calculations performed with the FE-model as described in chapter 4, and not to determine which analysis method is the closest to reality since this is already been done by Taris [10]. Determination of the most valid analysis method will be done in chapter 6 for a case including the leaning column effect.

Overall it can be stated that the possible difference between the results of the validation study and Taris for all analysis methods can be explained by the different way of modeling the cross-section, as can be seen in Figure 34. The difference between the models with and without overlap in the cross-section has led to a different moment of inertia and area, resulting in (minimal) differences in magnitudes of the imperfections. These differences are shown in Table 5.

Table 5: Differences for the properties and magnitudes of the imperfections

	Taris	Author's validation study
<i>Cross-section:</i>	$A = 22350$ $I_y = 217231 \cdot 10^4 \text{ mm}^4$ $I_z = 2938 \cdot 10^4 \text{ mm}^4$	$A = 21900$ $I_y = 210817 \cdot 10^4 \text{ mm}^4$ $I_z = 2936 \cdot 10^4 \text{ mm}^4$
<i>Imperfection:</i>	GMNIA I: In-plane = 28,9 mm GMNIA IIa: In-plane = 28,9 mm Out-of-plane = 45.1 mm GMNIA IIb: In-plane = 28,9 mm Out-of-plane = 22.8 mm GMNIA IIIa and IIIb: In-plane = 23,5 mm Out-of-plane = 6.76 mm	GMNIA I: In-plane = 28,9 mm GMNIA IIa: In-plane = 28,9 mm Out-of-plane = 44.68 mm GMNIA IIb: In-plane = 28,9 mm Out-of-plane = 22.53 mm GMNIA IIIa and IIIb: In-plane = 23,35 mm Out-of-plane = 6.7 mm

The results of the ultimate LPF for the various analyses are given in Table 6 and Figure 46. Here can be seen that the differences are minimal, except for GMNIA IIIb. The minimal differences for the majority of the analyses can be explained by the difference in cross-section properties as explained above, and the fact that the in-plane imperfection is induced by equivalent forces, instead of the normalized matrix of displacements, as Taris did. The larger difference for GMNIA IIIb is harder to explain. When we compare IIIa (coarse mesh) and IIIb (fine mesh) it is expected that the finer mesh will result in a lower ultimate LPF, but instead, the value is 10% higher. Figure 43 shows that the load-displacement path for GMNIA IIIa and GMNIA IIIb of Taris (solid line with ditto color) leaps down, where the load-displacement path for GMNIA IIIb in this study leaps up. The reason for that is unknown.

As discussed before, GMNIA II and GMNIA IV are unique results and can not be compared to the results of Taris to validate the model. Nevertheless, the results of Table 6 indicate that GMNIA II and GMNIA IV give similar results.

Table 6: Comparison of the various analyses

	Ultimate LPF							
	LBA	GMNIA I	GMNIA II	GMNIA IIa	GMNIA IIb	GMNIA IIIa	GMNIA IIIb	GMNIA IV
Taris	2.56	2.44	-	1.47	1.63	1.81	1.78	-
Author	2.61	2.52	1.65	1.48	1.64	1.82	1.99	1.73
Difference $\frac{T-X}{T}$	1.69%	3.07%	-	0.44%	0.66%	0.38%	10.60%	-

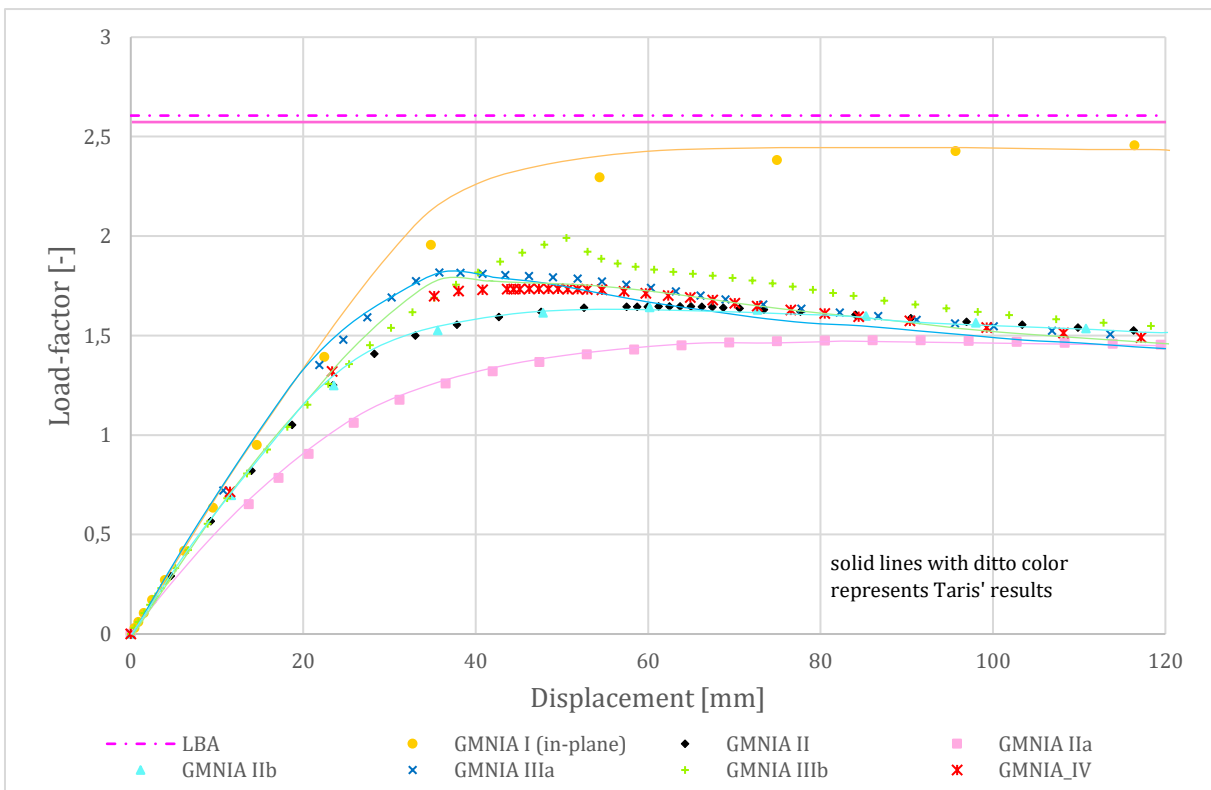


Figure 46: Load-displacement diagram with the results of all the analyses

6 CASE STUDY SWAY FRAME INCLUDING LEANING COLUMN

6.1 INTRODUCTION

The next step is to perform the various analyses once again, but now for a frame including the leaning column effect. This will be elaborated in this chapter. In section 6.5, the ultimate resistance load of the frame will be determined according to the design rules of the NEN 6771 and the EN 1993-1-1. The goal of this chapter is to determine which of these various analyses reflect reality in the best way for a frame including the leaning column effect and how this reflects the design codes.

The frame that will be studied is an unbraced fully lateral supported steel sway frame with a leaning column. The leaning column effect is included by adding a beam and column, where the beam is (and must be) pin-ended connected to the stabilizing frame, as shown in Figure 3. The fully lateral support condition is met by preventing the displacement of the webs of all members in the Z-direction, as discussed in section 4.3.3. The frame is loaded with a vertical point load F applied at the top of every column. The dimensions of the frame and the cross-sections are shown in Figure 47. The mechanical scheme that is used in this case study represents the execution of the frame that is the most vulnerable for the leaning column effect, shown in Figure 48.

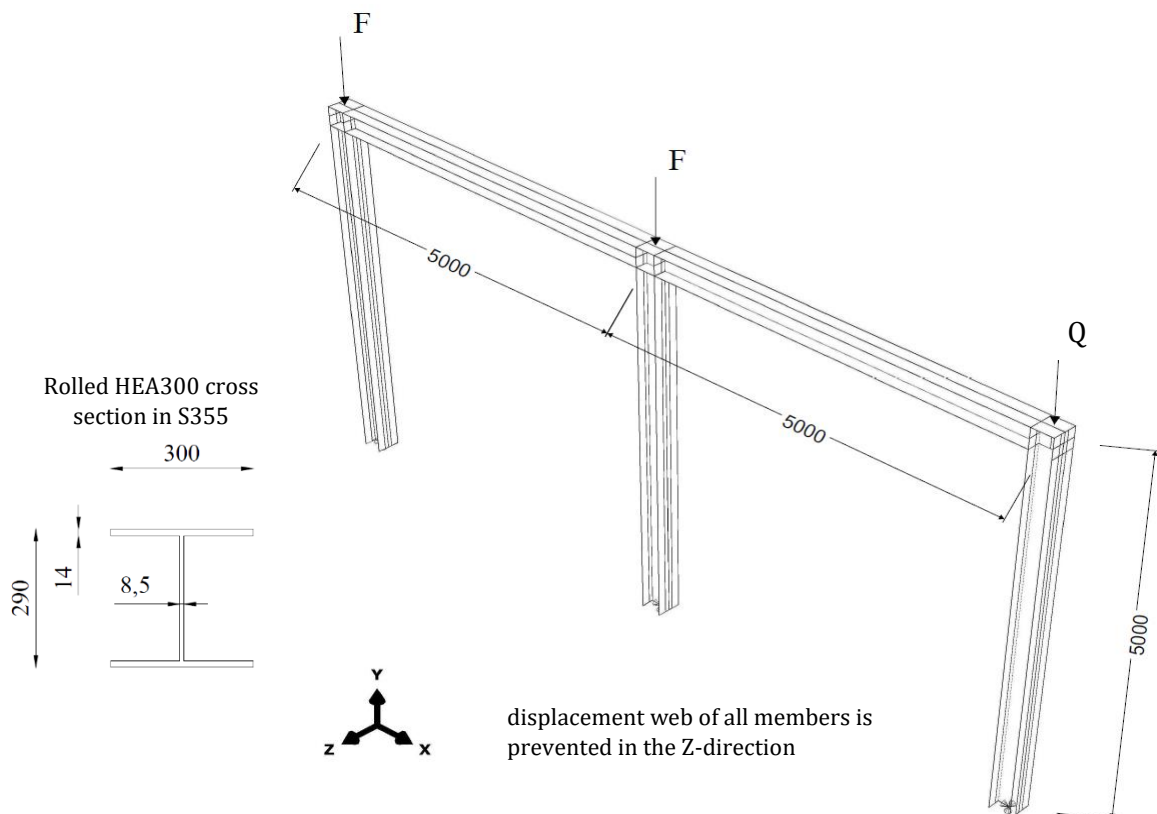


Figure 47: Three-dimensional illustration of the frame including the leaning column effect

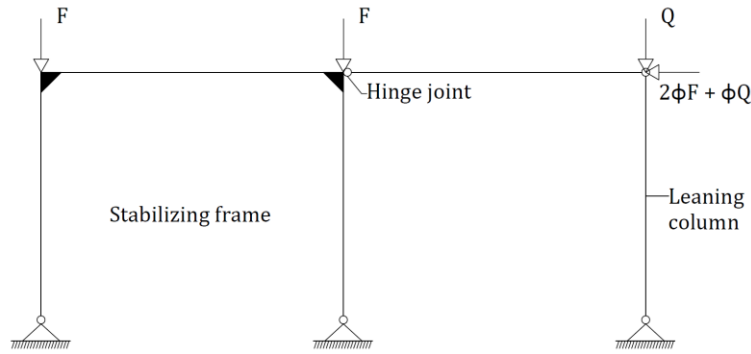


Figure 48: Mechanical scheme for the leaning column problem considered in the case study

In contrast to chapter 5, elastic-plastic material properties are used with linear strain hardening to reflect reality even more. True stress-strain behavior, can not be used due to a lack of material test data. The material that is used has a yield-strength of 355 N/mm² and an ultimate tensile strength of 490 N/mm². It turned out that the results are not affected by the linear strain hardening, since the problem that is considered is an instability problem.

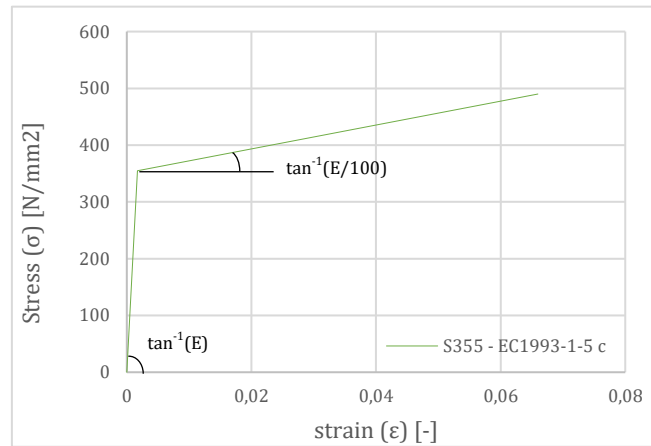


Figure 49: Engineering stress-strain diagram for elastic-plastic material with linear strain-hardening behavior for steel quality S355 according to EN 1993-1-5

6.2 ANALYSIS METHOD AND IMPERFECTIONS

The structure in this case study is fully restricted in the out-of-plane direction and only included in-plane imperfections. The imperfections are shaped and sized according to section 3.3. Table 7 shows how the imperfections are determined and how it is induced on the frame.

Table 7: Imperfections that are used in the various analyses in the case study

ANALYSES	IMPERFECTIONS	RESIDUAL STRESSES
LBA	-	-
GMNIA I	Sway imperfection according to EN 1993-1-1 Clause 5.3.2(3) induced with equivalent forces	NO
GMNIA II	First eigenmode sized according to EN 1993-1-1 Clause 5.3.2(11) (η_{init}) induced with the normalized matrix of displacements	NO
GMNIA III	First eigenmode sized as $L_{cr}/1000$ induced with the normalized matrix of displacements	YES
GMNIA IV	First three eigenmodes sized as $A_j = P_j \cdot F \cdot h$ induced with the normalized matrix of displacements	YES

6.3 MESH REFINEMENT STUDY AND THE CORRESPONDING RESIDUAL STRESSES

In section 5.3 only a smaller mesh refinement study is elaborated. The reason for that is that the results of chapter 5 are only for validation purposes, and not for obtaining actual results regarding this research. For the case study including the leaning column, the mesh is even more important. The mesh size used in the model is determinative of the results that will be obtained from the FE-Model. With a mesh-refinement study, the mesh is found where the result-difference is found acceptable compared to the finest mesh. Also regarding residual stresses, the mesh size plays an important role. A finer mesh is needed to precisely model the configuration of the residual stresses as discussed in section 3.3.3. The model should be modeled in such a way that the mesh size is adjustable in the region of the intersection of the beam with the column.

Also for the case study, the mesh is studied with an elastic LBA and a plastic GMNIA, since the deformation of the frame is hard to compare. It turned out that the starting mesh (314x145x50) already gave reasonable results. This allowed smaller divide steps to obtain reasonable results and therefore different combinations of mesh sizes are studied, resulting in various mesh element shapes. The results of the mesh refinement study are shown in Table 8 and Table 9. The mesh refinement study resulted in a mesh of 78.5x32.75x25, with only 0.11%, 0.33, and 0.42% differences compared to the finest mesh. The mesh size direction of the coding a, b and c are shown in Figure 50.

Table 8: Results for the elastic mesh-refinement study for the case study

axbxc	Number of elements	Rel. mesh density $\frac{x}{50 \times 145 \times 314}$	Critical force amplifier (α_{cr})	CPU time	% Error $\frac{26.2 \times 22.2 \times 25 - x}{26.2 \times 22.2 \times 25}$
	[-]	[-]	[-]	[s]	(%)
26.2x22.2x25	22370	23.01	1.7877	440.3	0.00%
39.25x32.75x25	16286	16.75	1.7882	262.5	-0.03%
78.5x32.75x25	8536	8.78	1.7897	98.8	-0.11%
78.5x65.5x25	7248	7.46	1.7903	84.8	-0.15%
52.33x49.12x50	7058	7.26	1.7947	102.3	-0.39%
157x32.75x25	4692	4.83	1.7926	24.0	-0.27%
78.5x65.5x50	4128	4.25	1.796	42.7	-0.46%
157x65.5x25	3832	3.94	1.7932	21.0	-0.31%
157x65.5x50	2176	2.39	1.7984	12.4	-0.60%
314x65.5x50	1216	1.25	1.804	6.9	-0.91%
314x145x50	972	1	1.8074	5.6	-1.10%

Table 9: Results for the plastic mesh-refinement study for the case study

axbxc	Rel. mesh density $\frac{X}{50 \times 145 \times 314}$	Reaction force (RF)	Applied force (AF)	CPU time	% Error $\frac{6.25 \times 6.25 \times 6.25 - X}{6.25 \times 6.25 \times 6.25}$	
	[-]	[kN]	[kN]	[s]	RF (%)	AF (%)
26.2x22.2x25	23.01	1440.2	1324.2	2038.8	0.00%	0.00%
39.25x32.75x25	16.75	1441.3	1325.6	1180.0	-0.08%	-0.11%
78.5x32.75x25	8.78	1444.5	1329.7	611.8	-0.30%	-0.42%
78.5x65.5x25	7.46	1444.8	1329.1	507.3	-0.32%	-0.37%
52.33x49.12x50	7.26	1427.8	1309.1	483.8	0.86%	1.14%
157x32.75x25	4.83	1450.0	1336.5	340.1	-0.68%	-0.93%
78.5x65.5x50	4.25	1431.5	1312.1	283.1	0.60%	0.92%
157x65.5x25	3.94	1449.6	1335.0	276.6	-0.66%	-0.81%
157x65.5x50	2.39	1442.7	1315.9	155.9	-0.17%	0.63%
314x65.5x50	1.25	1457.9	1328.2	94.2	-1.23%	-0.31%
314x145x50	1	1453.5	1325.9	74.6	-0.92%	-0.13%

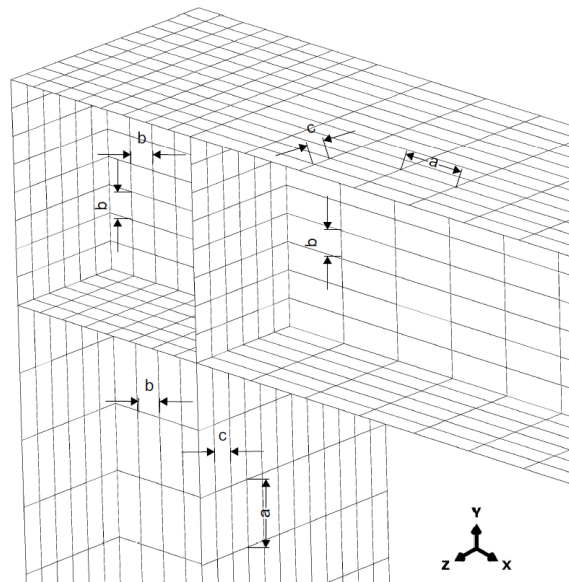


Figure 50: Mesh-size coding illustrated on the left column-beam joint of the leaning column problem

In the case study, HEA300 cross-sections are used for the members of the frame. The HEA300 cross-section is fabricated with a hot-rolling process. Residual stresses develop due to differential cooling after hot rolling. Figure 51 shows on the left the actual residual stress for a HEA300. The patterns for the FE-model are based on the mesh size 78.5x32.75x25 and will be used in analysis methods GMNIA III and GMNIA IV. The corresponding residual stress pattern is shown on the right of Figure 51. In contrast to chapter 5, there is no distinction made between the coarse or fine mesh and therefore only one mesh size is used for residual stresses.

Vogel et al. [8] do not advise how to deal with residual stresses for material that differ from the S235 steel quality. It is expected that the residual stresses for S355 are lower than $0.5f_y$. In the process, calculations were processed with $0.375f_y$, to see the influence of lower residual stresses. It turned out that the results only differ 1-2% and for that reason, the rule of $0.5f_y$ will be maintained.

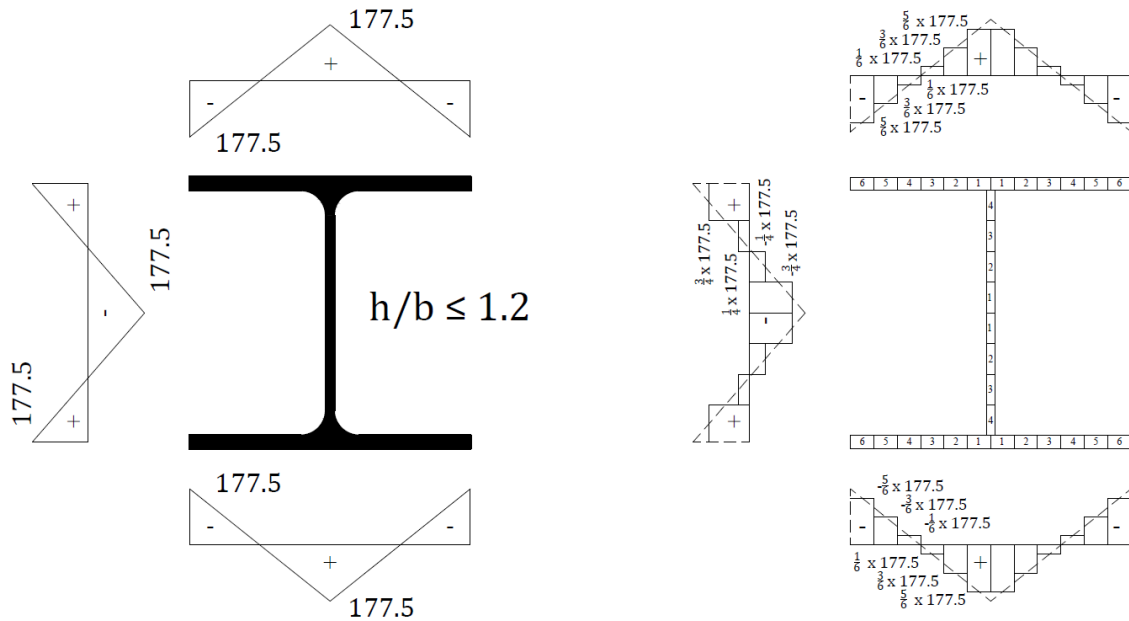


Figure 51: Residual stress pattern for the actual cross-section (left), and the FE-model mesh (right)

6.4 NUMERICAL RESULTS OF THE VARIOUS ANALYSES

6.4.1 LBA

As mentioned before, the study of this chapter only takes in-plane displacements into account. This is achieved by a model that is laterally supported at the webs in the Z-direction. This results in only in-plane eigenmodes, where the first eigenmode is the critical sway-mode eigenmode. The frame as shown in Figure 47 is loaded with $F=Q=1000$ kN. The critical eigenmode for the frame is shown in Figure 52. The critical eigenmode results in a load factor of $\alpha_{cr} = 1.7898$, also shown in Figure 52. This results in an ultimate load of $F_{ult,LBA} = F_{cr} = 1789.8$ kN. The corresponding section moment at the critical cross-section is $EI|\eta_{cr}''|_{max} = 2.575 \cdot 10^6$ Nmm, where the critical cross-section is located on the column directly under the lower stiffener and is shown in Figure 53. Due to explicitly modeling shell elements with the real connections, the length of the elements will decrease compared to the beam elements. This is because the columns and connections are separately modeled when using shell elements, resulting in a somewhat lower $EI|\eta_{cr}''|_{max}$. However, in the progress, it has been experienced that this difference does not influence the results.

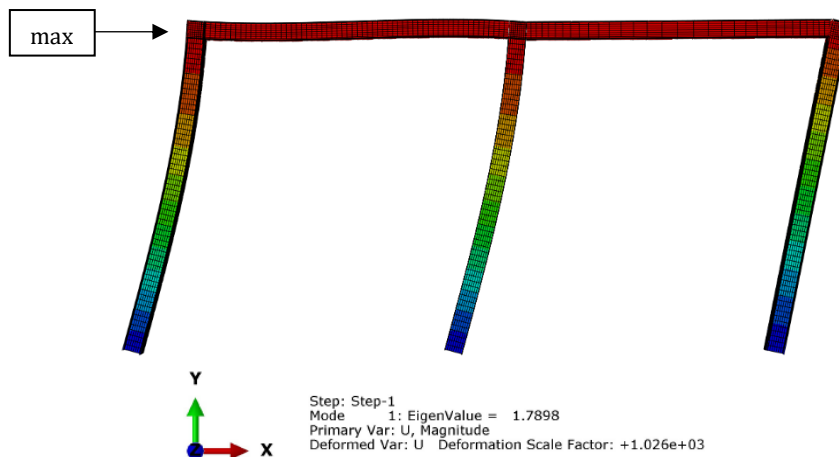


Figure 52: Critical eigenmode for a frame with a leaning column

Figure 53: Resultant of section-moment $EI|\eta_{cr}''|_{max}$ for the eigenmode of Figure 52, plotted on the undeformed state

6.4.2 GMNIA I according to EN 1993-1-1 Clause 5.3.2(3)

The buckling shape found in Figure 52 follows a sway mode, where each column additionally deforms with a bow imperfection. For that reason both sway- and bow imperfections of the EN 1993-1-1 Clause 5.3.2(3) are induced on the frame, and are determined as the following:

Sway imperfection:

$$\phi_0 = 1/200$$

$$\alpha_h = \frac{2}{\sqrt{h}} = \frac{2}{\sqrt{5}} = 0.894, \text{ where } \frac{2}{3} \leq 0.894 \leq 1$$

$$\alpha_m = \sqrt{0.5 \left(1 + \frac{1}{m}\right)} = \sqrt{0.5 \left(1 + \frac{1}{3}\right)} = 0.816$$

$$\phi = \phi_0 \cdot \alpha_h \cdot \alpha_m = \frac{1}{200} \cdot 0.894 \cdot 0.816 = 3.647 \cdot 10^{-3}$$

$$\Delta = 3.647 \cdot 10^{-3} \cdot 5000 = 18.3 \text{ mm}$$

Bow imperfection:

A HEA300 profile in S355 gives a section classification of 3 for outstand flanges loaded in compression, so the elastic section modulus has to be used.

$$I = \frac{1}{12} \cdot 8.5 \cdot 262^3 + 2 \cdot \left(\frac{1}{12} \cdot 300 \cdot 14^3 + 14 \cdot 300 \cdot 138^2 \right) = 1728.5 \cdot 10^5 \text{ mm}^4$$

$$W_{el} = \frac{1728.5 \cdot 10^4}{145} = 119.2 \cdot 10^4 \text{ mm}^3$$

$$M_{Rk} = f_y \cdot W_{el} = 355 \cdot 119.2 \cdot 10^4 = 423.2 \cdot 10^6 \text{ Nmm}$$

$$N_{Rk} = f_y \cdot A = 355 \cdot 10627 = 3772.6 \cdot 10^3 \text{ N}$$

$$F_{cr} = 1789.8 \cdot 10^3 = 1789.8 \cdot 10^3 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr}}} = \sqrt{\frac{N_{Rk}}{F_{cr}}} = \sqrt{\frac{3772.6 \cdot 10^3}{1789.8 \cdot 10^3}} = 1.45$$

$\alpha = 0.34$ according to table 1 in appendix A

$$e_0 = \alpha(\bar{\lambda} - 0.2) \frac{M_{Rk}}{N_{Rk}} = 0.34 \cdot (1.45 - 0.2) \frac{423.2 \cdot 10^6}{3772.6 \cdot 10^3} = 47.7 \text{ mm}$$

The Sway imperfection Δ and bow imperfection e_0 are induced on the structure as an equivalent force as shown in Figure 14. This GMNIA results in an ultimate load of $F_{ult,GMNIA I} = 1338.3$ kN. The load-displacement path corresponding to this analysis is shown in Figure 54, where each dot is a converged equilibrium state. The displacement taken is the lateral displacement at the left top node. The figure also includes the F_{cr} obtained from the LBA.

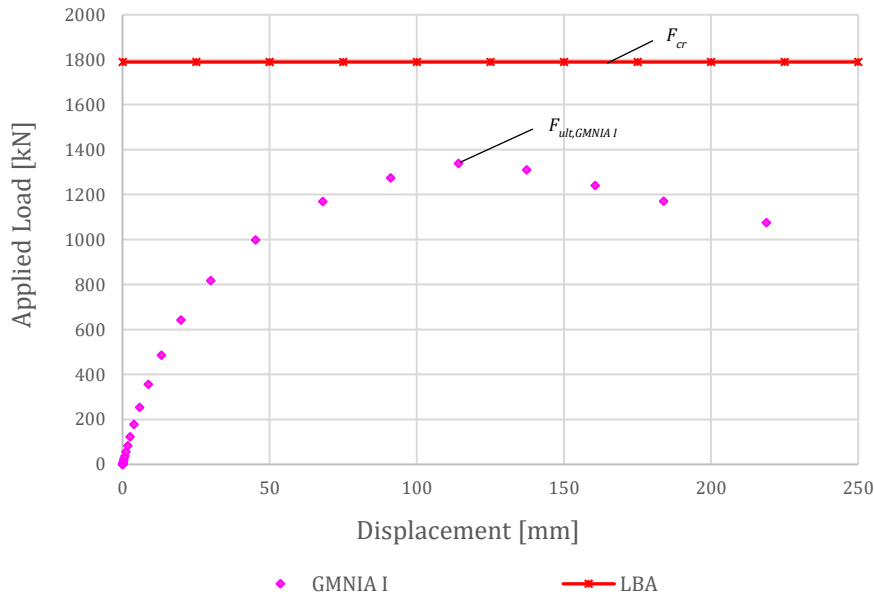


Figure 54: Load displacement diagram for GMNIA I, including the buckling load from the LBA

6.4.3 GMNIA II according to EN 1993-1-1 clause 5.3.2(11)

The imperfection for GMNIA II is induced with a scale-factor (η_{init}) to size the normalized matrix of displacements of the critical eigenmode. The scale-factor is determined as the following:

$$M_{Rk} = 423.2 \cdot 10^6 \text{ Nmm according to section 6.4.2}$$

$$N_{Rk} = 3772.6 \cdot 10^3 \text{ N according to section 6.4.2}$$

$$\bar{\lambda} = 1.45 \text{ according to section 6.4.2}$$

$$\alpha = 0.34 \text{ according to section 6.4.2}$$

$$\phi = 0.5[1 + \alpha(\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 0.5[1 + 0.34 \cdot (1.45 - 0.2) + 1.45^2] = 1.766$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = \frac{1}{1.766 + \sqrt{1.766^2 - 1.45^2}} = 0.36$$

$$e_0 = \alpha(\bar{\lambda} - 0.2) \frac{M_{Rk}}{N_{Rk}} \cdot \frac{1 - \chi^{\bar{\lambda}^2}}{1 - \chi^2} = 0.34 \cdot (1.45 - 0.2) \frac{423.2 \cdot 10^6}{3772.6 \cdot 10^3} \cdot \frac{1 - \frac{0.36^{1.45^2}}{1}}{1 - 0.36^{1.45^2}} = 47.7 \text{ mm}$$

$$EI|\eta_{cr}''|_{max} = 2.575 \cdot 10^6 \text{ according to Figure 53}$$

$$\eta_{init} = e_0 \cdot \frac{F_{cr}}{EI |\eta_{cr}|_{max}} \eta_{cr} = 47.7 \cdot \frac{1789.8 \cdot 10^3}{2.575 \cdot 10^6} = 33.2 [-]$$

The magnitude of the scale-factor η_{init} , which is used to size the normalized matrix of displacements of the critical eigenmode, is $\eta_{init} = 33.2$. This GMNIA results in an ultimate load of $F_{ult,GMNIA II} = 1507.0$ kN. The load-displacement path corresponding to this analysis is shown in Figure 55, where each dot is a converged equilibrium state. The displacement taken is the lateral displacement at the left top node.

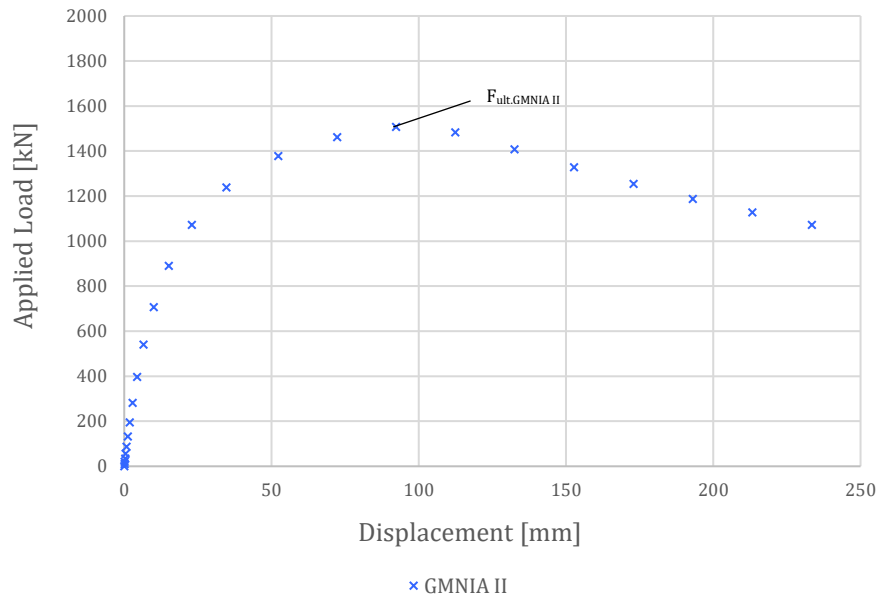


Figure 55: Load displacement diagram for GMNIA II

6.4.4 GMNIA III according to Vogel et al.

For GMNIA III, the magnitude of the imperfections is derived according to Vogel et al. [8], as described in section 3.3.3. The residual stress pattern that is plotted on the structure is shown in section 6.3. The determination of the magnitude of the imperfection to scale the normalized matrix of displacements for the critical eigenmode is the following:

$$N_{Ed} = 1000 \cdot 10^3 \text{ N}$$

$$I = 1728.5 \cdot 10^5 \text{ mm}^4 \text{ according to section 6.4.2}$$

$$\alpha_{cr} = 1.7898 \text{ according to Figure 52}$$

$$L_{cr} = \pi \cdot \sqrt{\frac{E \cdot I}{\alpha_{cr} \cdot N_{Ed}}} = \pi \cdot \sqrt{\frac{210000 \cdot 1.7285 \cdot 10^8}{1.7898 \cdot 1000000}} = 14148 \text{ mm}$$

$$\frac{L_{cr}}{1000} = \frac{14148}{1000} = 14.15 \text{ mm}$$

The formula $L_{cr}/1000$ gives the sway imperfection at the middle of the buckling length. The middle of the buckling length, however, does not match with the height of the frame. To obtain the imperfection at the height of the frame, a geometrical correction has to be performed, as described in section 3.3.3. The geometrical correction can be derived by the following:

$$P = r - 14.15$$

$$r^2 = P^2 + \left(\frac{14148}{2}\right)^2$$

$$(P + 14.15)^2 = P^2 + \left(\frac{14148}{2}\right)^2$$

$$P^2 + 28.3P + 200.2 = P^2 + \left(\frac{14148}{2}\right)^2, \text{ gives } P = 1768459.8 \text{ mm}$$

$$r = 1768474 \text{ mm}$$

$$x = \frac{14148}{2} - 5000 = 2073.9 \text{ mm}$$

$$y = \sqrt{1768474^2 - 2073.9^2} = 1768472.7 \text{ mm}$$

$$\Delta = 1768472.7 - 1768459.8 = 12.93 \text{ mm (without rounding off)}$$

Value $\Delta = 12.93$ is the magnitude to scale the normalized matrix of displacements of the critical eigenmode. This GMNIA results in an ultimate load of $F_{ult,GMNIA III} = 1604.9$ kN. The load-displacement path corresponding to this analysis is shown in Figure 56, where each dot is a converged equilibrium state. The displacement taken is the lateral displacement at the left top node.

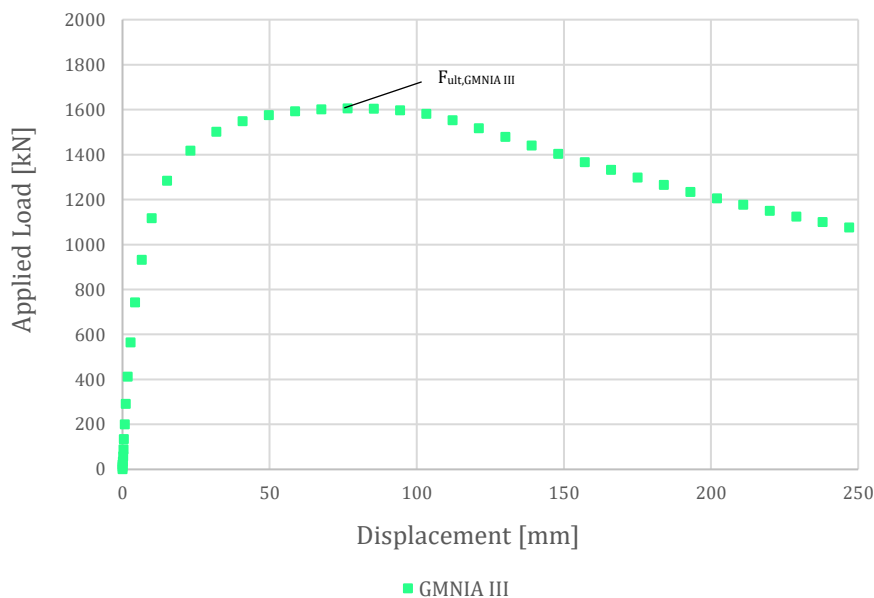


Figure 56: Load displacement diagram for GMNIA III

6.4.5 GMNIA IV according to Shayan et al. and Vogel et al.

Analysis method GMNIA IV makes use of imperfections based on various eigenmodes with scale-factors according to Shayan et al. [9], and residual stresses according to Vogel et al. [8], as described in section 3.3.4. This section also described that the use of three eigenmodes already gives valid results. However, since we are studying a leaning column problem, all the three eigenmodes should be side-sway eigenmode. It has been experienced in the process that otherwise not the lowest ultimate load that the frame can carry is found. For the frame considered in this case-study, only two eigenmodes have this side-sway eigenmode, mode 1 and mode 64, and are shown in Figure 57. The side-sway eigenmodes deform in the opposite direction. This will be corrected in the imperfection amplitude factor.

The magnitude of the normalized matrix of displacements are based on Table 1 and given as the following:

$$P_1 = 0.782 \text{ according to Table 1}$$

$$P_2 = 0.218 \text{ according to Table 1}$$

$$F_{shayan} = 0.001566 \text{ according to Table 1}$$

$$h = 5000 \text{ mm}$$

$$A_1 = -(P_1 \cdot F_{shayan} \cdot h) = -(0.782 \cdot 0.001566 \cdot 5000) = -6.12$$

$$A_2 = P_2 \cdot F_{shayan} \cdot h = 0.218 \cdot 0.001566 \cdot 5000 = 1.71$$

Values A_1 and A_2 are the magnitudes to scale the normalized matrix of displacements of the two eigenmodes, modes 1 and 64 respectively. This GMNIA results in an ultimate load of $F_{ult,GMNIA IV} = 1671.9$ kN. The load-displacement path corresponding to this analysis is shown in Figure 58, where each dot is a converged equilibrium state. The displacement taken is the lateral displacement at the left top node.

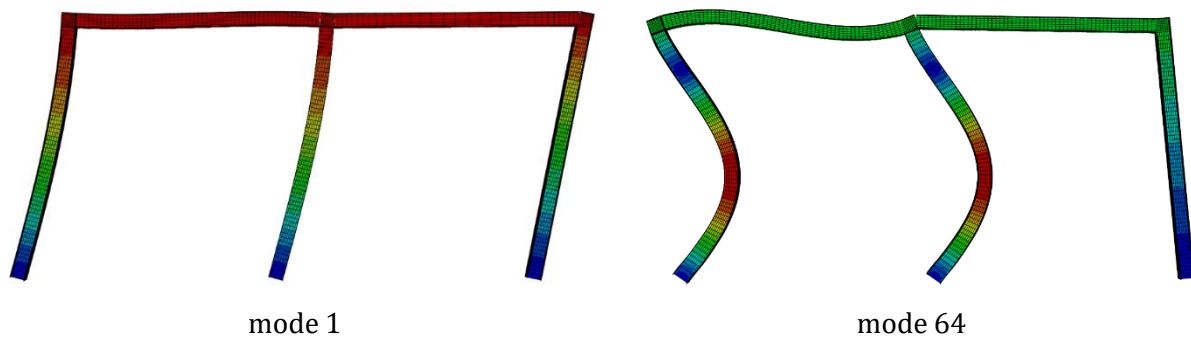


Figure 57: The first two side-sway eigenmodes for the normalized matrix of displacements that are used on the structure in GMNIA IV

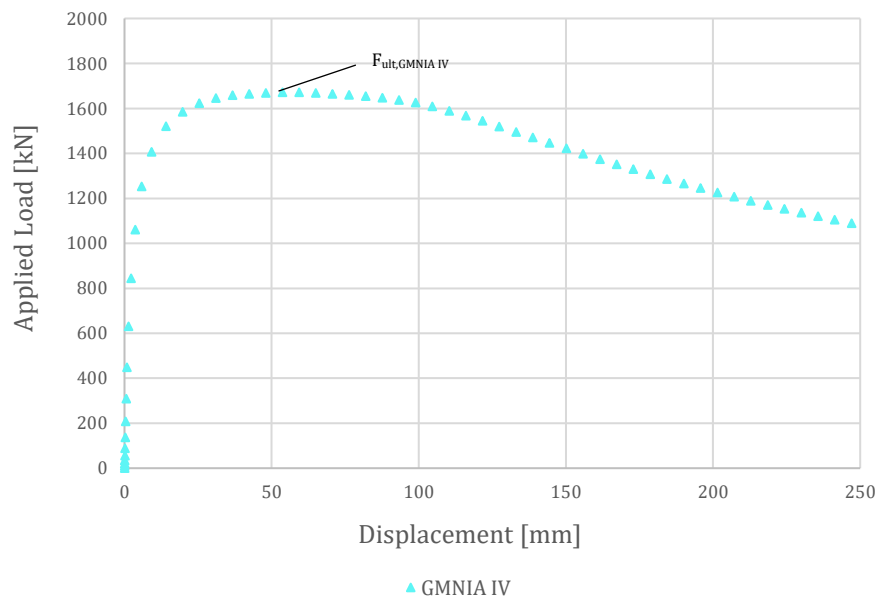


Figure 58: Load displacement diagram for GMNIA IV

6.5 ULTIMATE RESISTANCE LOADS ACCORDING TO THE CODES

6.5.1 NEN 6771

The assessment criteria for a column loaded in compression and bending according to the NEN 6771 is given by equation (2) in section 1.1.2.1 and is the following:

$$\frac{N_{Ed}}{N_{Rd}} + \frac{\alpha_{cr}}{\alpha_{cr} - 1} \cdot \frac{F_{tot} \cdot e_0 + M_{Ed}}{M_{Rd}} \leq 1$$

To determine the ultimate resistance load, the axial load N_{Ed} in the equation above should be increased until unity is reached. This will result in $F_{ult,DR,NEN}$, where the elaboration of $F_{ult,DR,NEN}$ is given below:

$$\alpha_{cr} = 1.7898, \text{ according to section 6.4.1.}$$

To determine L_{cr} , α_{cr} is used from the LBA from section 6.4.1. This is the method that reflects reality in the best way. Section 6.7 will discuss the influence of this choice.

$$L_{cr} = \pi \cdot \sqrt{\frac{E \cdot I}{\alpha_{cr} \cdot N_{Ed}}} = \pi \cdot \sqrt{\frac{210000 \cdot 1.7285 \cdot 10^8}{1.7898 \cdot 1000 \cdot 10^3}} = 14148 \text{ mm}$$

$$i = \sqrt{\frac{I}{A}} = \sqrt{\frac{1.7285 \cdot 10^8}{10627}} = 127.53 \text{ mm}$$

$$\lambda_e = \pi \cdot \sqrt{\frac{E}{f_{y,d}}} = \pi \cdot \sqrt{\frac{210000}{355}} = 76.41$$

$$\lambda = \frac{L_{cr}}{i} = \frac{14148}{127.53} = 110.93$$

$$\lambda_{rel} = \frac{\lambda}{\lambda_e} = \frac{111.61}{76.41} = 1.45$$

$$M_{Rd} = \frac{f_{y,d} \cdot W_{el}}{\gamma_0} = \frac{355 \cdot 1.192 \cdot 10^6}{1} = 423.2 \cdot 10^6 \text{ Nmm}$$

$$N_{Rd} = \frac{f_{y,d} \cdot A}{\gamma_0} = \frac{355 \cdot 10627}{1} = 3772.6 \cdot 10^3 \text{ N}$$

$$\alpha_k = 0.34, \text{ according to table 25 of the NEN 6770}$$

$$\lambda_0 = 0.20, \text{ according to table 25 of the NEN 6770}$$

$$e_0 = \alpha_k \cdot (\lambda_{rel} - \lambda_0) \cdot \frac{M_{Rd}}{N_{Rd}} = 0.34 \cdot (1.45 - 0.2) \cdot \frac{423.2 \cdot 10^6}{3772.6 \cdot 10^3} = 47.7 \text{ mm}$$

$$\alpha_{cr} = \frac{F_{cr}}{N_{Ed}} = \frac{1789.8 \cdot 10^3}{N_{ult}}$$

$$\phi_0 = \frac{1}{250}, n_s = 1, n_k = 3$$

$$k_1 = \sqrt{0.2 + \frac{1}{n_s}}, \text{ but } \leq 1 \text{ gives } \sqrt{0.2 + \frac{1}{1}} = 1.095 = 1$$

$$k_2 = \sqrt{0.5 + \frac{1}{n_s}}, \text{ but } \leq 1 \text{ gives } \sqrt{0.5 + \frac{1}{3}} = 0.913$$

$$\phi = \phi_0 \cdot k_1 \cdot k_2 = \frac{1}{250} \cdot 1 \cdot 0.913 = 3.647 \cdot 10^{-3}$$

$$\phi N_{Ed} = \phi N_{ult} = 3.647 \cdot 10^{-3} \cdot N_{ult}$$

To make the results comparable, the Eurocode series is leading. The bending moment factor is taken according to the Eurocode (0.9) instead of the NEN 6771 (0.85):

$$M = 0.9 \cdot M_{Ed} = 0.9 \cdot \frac{3 \cdot \phi F \cdot h}{2} = \frac{3 \cdot \phi F_{ult} \cdot h}{2} = 0.9 \cdot \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot N_{ult} \cdot 5000}{2}$$

According to Figure 6, F_{tot} is equal to $F_{tot} = \frac{3}{2} N_{Ed}$, resulting in $F_{tot,ult} = \frac{3}{2} F_{ult}$

$$\frac{N_{ult}}{3772.6 \cdot 10^3} + \frac{\frac{1789.8 \cdot 10^3}{N_{ult}}}{\frac{1789.8 \cdot 10^3}{N_{ult}} - 1} \cdot \frac{\frac{3}{2} F_{ult} \cdot 47.7 + 0.9 \cdot \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot N_{ult} \cdot 5000}{2}}{423.2 \cdot 10^6} = 1$$

Solving the equation above results in an ultimate resistance load of:

$$N_{ult,DR,NEN,LBA} = 1131.6 \text{ kN}$$

It should be noted that the leaning column effect now is taken into account twice. First with α_{cr} , since the leaning column is modeled, and later with F_{tot} .

6.5.2 EN 1993-1-1

The assessment criteria for a column loaded in compression and bending according to the EN 1993-1-1 is given by equation (25) in section 1.1.2.2 and is the following:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}}$$

To determine the ultimate resistance load, N_{Ed} in the equation above should be increased until unity is reached. This will result in $F_{ult,DR,EC}$, where the elaboration of $F_{ult,DR,EC}$ is given below. To determine L_{cr} , α_{cr} is used from the LBA from chapter 6.4.1.

$$F_{cr} = 1789.8 \cdot 10^3 \text{ N, according to section 6.4.2}$$

$$\bar{\lambda} = 1.45, \text{ according to section 6.4.2}$$

$$\Phi = 1.766, \text{ according to section 6.4.3}$$

$$\chi = 0.36, \text{ according to section 6.4.3}$$

$$N_{Rk} = 3772.6 \cdot 10^3 \text{ N according to section 6.4.2}$$

$$N_{Rd} = \frac{N_{Rk}}{\gamma_0} = \frac{3772.6 \cdot 10^3}{1} = 3772.6 \cdot 10^3 \text{ N}$$

$M_{Rk} = 423.2 \cdot 10^6$ Nmm according to section 6.4.2

$$M_{Rd} = \frac{M_{Rk}}{\gamma_0} = \frac{423.2 \cdot 10^6}{1} = 423.2 \cdot 10^6 \text{ Nmm}$$

$\phi = 3.647 \cdot 10^{-3}$ according to section 6.4.2

$C_m = 0.9$ according to table 7 in appendix A

$$\phi N_{Ed} = \phi N_{ult} = 3.647 \cdot 10^{-3} \cdot N_{ult}$$

$$C_m \cdot M_{Ed} = 0.9 \cdot \frac{3 \cdot \phi F \cdot h}{2} = \frac{3 \cdot \phi N_{ult} \cdot h}{2} = 0.9 \cdot \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot N_{ult} \cdot 5000}{2}$$

$$k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}} = \frac{1}{1 - \frac{0.36 \cdot N_{ult}}{1789.8 \cdot 10^3}}$$

$$\frac{N_{ult}}{0.357 \cdot 3772.6 \cdot 10^3} + \frac{1}{1 - \frac{0.36 \cdot N_{ult}}{1789.8 \cdot 10^3}} \cdot \frac{0.9 \cdot \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot N_{ult} \cdot 5000}{2}}{423.2 \cdot 10^6} = 1$$

Solving the equation above results in an ultimate resistance load of:

$$N_{ult,DR,EC,LBA} = 1230.6 \text{ kN}$$

6.5.3 Modified EC1991-1-1

In the following derivation, the leaning column effect is included in the assessment criteria for a column loaded in compression and bending according to the EN 1993-1-1, as derived in section 2.1. The equation is given by equation (43) and is the following:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \left(\bar{\lambda}^2 - \frac{1}{\chi} \right) (\chi - 1) \frac{F_{lean}}{N_{Rd}} + k \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}}$$

To determine the ultimate resistance load, N_{Ed} in the equation above should be increased until unity is reached. This will result in $N_{ult,DR,modEC}$, where the elaboration of $N_{ult,DR,modEC}$ is given below:

To determine L_{cr} , α_{cr} is used from the LBA from chapter 6.4.1.

$F_{cr} = 1789.8 \cdot 10^3$ N, according to section 6.4.2

$\bar{\lambda} = 1.45$, according to section 6.4.2

$\phi = 1.766$, according to section 6.4.3

$\chi = 0.36$, according to section 6.4.3

$\phi = 3.647 \cdot 10^{-3}$ according to section 6.4.2

$C_m = 0.9$ according to table 7 in appendix A

$$\phi N_{Ed} = \phi F_{ult} = 3.647 \cdot 10^{-3} \cdot F_{ult}$$

$N_{Rd} = 3772.6 \cdot 10^3$ N, according to section 6.5.2

$M_{Rd} = 423.2 \cdot 10^6$ Nmm, according to section 6.5.2

$$C_m \cdot M_{Ed} = \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot F_{ult} \cdot 5000}{2} \text{ according to section 6.5.2}$$

$$k = \frac{1}{1 - \frac{0.36 \cdot F_{ult}}{1789.8 \cdot 10^3}} \text{ according to section 6.5.2}$$

$$\begin{aligned} \frac{F_{ult}}{0.357 \cdot 3772.6 \cdot 10^3} + \frac{1}{1 - \frac{0.36 \cdot F_{ult}}{1789.8 \cdot 10^3}} \cdot \left(1.46^2 - \frac{1}{0.357}\right) (0.357 - 1) \frac{F_{lean}}{3772.6 \cdot 10^3} \\ + \frac{1}{1 - \frac{0.36 \cdot F_{ult}}{1789.8 \cdot 10^3}} \cdot \frac{0.9 \cdot \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot F_{ult} \cdot 5000}{2}}{423.2 \cdot 10^6} = 1 \end{aligned}$$

According to Figure 11, F_{lean} is equal to $F_{lean} = \frac{1}{2}Q = \frac{1}{2}F$, resulting in $F_{lean,ult} = \frac{1}{2}F_{ult}$

$$\begin{aligned} \frac{F_{ult}}{0.357 \cdot 3772.6 \cdot 10^3} + \frac{1}{1 - \frac{0.36 \cdot F_{ult}}{1789.8 \cdot 10^3}} \cdot \left(1.46^2 - \frac{1}{0.357}\right) (0.357 - 1) \frac{\frac{1}{2}F_{ult}}{3772.6 \cdot 10^3} \\ + \frac{1}{1 - \frac{0.36 \cdot F_{ult}}{1789.8 \cdot 10^3}} \cdot \frac{0.9 \cdot \frac{3 \cdot 3.647 \cdot 10^{-3} \cdot F_{ult} \cdot 5000}{2}}{423.2 \cdot 10^6} = 1 \end{aligned}$$

Solving the equation above results in an ultimate resistance load of:

$N_{ult,DR,modEC,LBA} = 1131.6$ kN, which is the same ultimate resistance load as the NEN 6771 (section 6.5.1). It should be noted that also for the modified EN 1993-1-1 equation, the leaning column effect is taken into account twice. First with α_{cr} , since the leaning column is modeled, and later with F_{lean} .

6.6 COMPARISON OF THE RESULTS

6.6.1 Differences and similarities

In this research, no actual tests are performed, meaning that the results cannot be compared to the actual real behavior of the structure. However, the GMNIA with residual stresses, GMNIA III and GMNIA IV, are assumed to be the closest to reality. These GMNIA's have the most realistic shape and magnitudes of the imperfections implemented and on top of that, they also have the residual stresses modeled. Ultimate loads $F_{ult,GMNIA III}$, and $F_{ult,GMNIA IV}$ are very close and only differ -4.1% of each other. This small difference strengthens the fact that the GMNIA III and GMNIA IV are close to reality since they both find (almost) the same result. This difference is also confirmed by the statement of Shayan et al. [9], which is that the use of only the first critical eigenmode (which is the case for GMNIA III) leads to conservative results. This confirmation is also the reason that from now all values that will be compared to the GMNIA, will be compared to GMNIA IV.

The difference with the ultimate load $F_{ult,GMNIA II}$ (-9.85%) can be described by the fact that the imperfections of the EN 1993-1-1 include equivalent material imperfections (residual stresses, etc.) which should cover all cases, and therefore GMNIA II is conservative. For the same reason GMNIA I is even more conservative since the bow and sway imperfections that are both included uses equivalent imperfections. This leads to a conservative $F_{ult,GMNIA I}$, which differ -19.95%. All the

load-displacement paths are shown in Figure 59, and the differences in ultimate loads are given in Table 10.

When we compare the ultimate loads obtained by the various analyses to the ultimate resistance loads obtained by the codes, it can be seen that the results of the EN 1993-1-1 lie under the results of the various analyses. The large (positive) differences as given in Table 11 indicate that the EN 1993-1-1 design rules are conservative and therefore safe for this frame. In Figure 59 and Table 10 also the ultimate resistance load for the NEN 6771 and modified EN 1993-1 is shown. Here it can be seen that the modified EN 1993-1-1 design rule from section 2.1 gives the same result as the NEN 6771. However, as already mentioned in section 6.5.1 and 6.5.3, the ultimate resistance loads of the NEN 6771 and the modified EN 1993-1-1 takes into account the leaning column effect twice, since the leaning column effect is also considered in the LBA and is for that reason a useless result. But since $N_{ult,NEN,LBA}$ and $N_{ult,modEC,LBA}$ are the most realistic value of the ultimate resistance load obtained by the NEN 6771 and the modified EN 1993-1-1 respectively they are still shown.

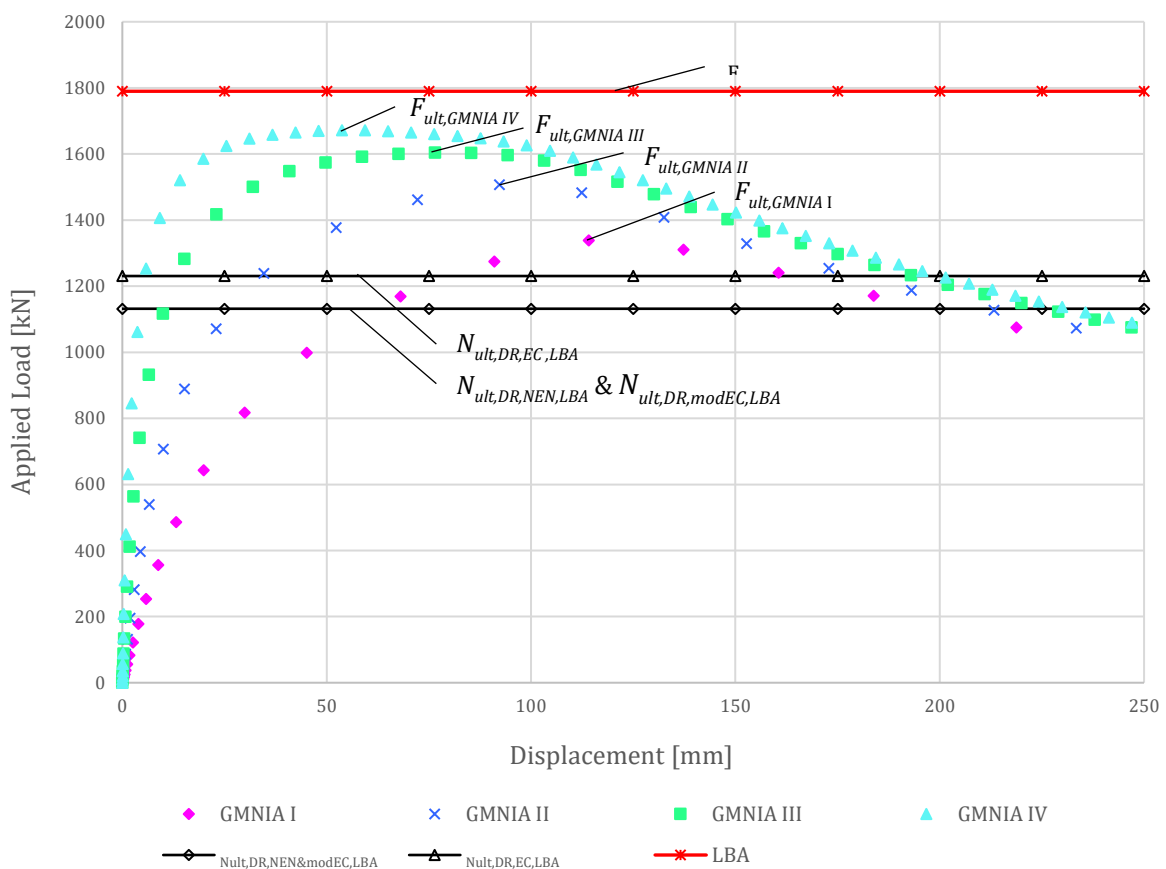


Figure 59: The load-displacement paths for all analyses including the ultimate resistance loads

Table 10: Differences ultimate (design-resistance) loads compared to the model closest to reality (GMNIA IV)

	Ultimate load (F_{ult}) in kN					
	$N_{ult,DR,NEN,LBA}$ & $N_{ult,DR,modEC,LBA}$	$N_{ult,DR,EC,LBA}$	$F_{ult,GMNIA I}$	$F_{ult,GMNIA II}$	$F_{ult,GMNIA III}$	$F_{ult,GMNIA IV}$
Result	1131.6 kN	1230.6 kN	1338.3 kN	1507.0 kN	1604.6 kN	1671.9 kN
Difference $\frac{x-GMNIA IV}{GMNIA IV}$	-32.31 %	-26.38 %	-19.95 %	-9.85 %	-4.06 %	0.00%

Table 11: Differences between the ultimate resistance loads compared to the ultimate loads

	Ultimate load (F_{ult}) in kN					
	$N_{ult,DR,NEN,LBA}$ & $N_{ult,DR,modEC,LBA}$	$N_{ult,DR,EC,LBA}$	$F_{ult,GMNIA I}$	$F_{ult,GMNIA II}$	$F_{ult,GMNIA III}$	$F_{ult,GMNIA IV}$
Result	1131.6 kN	1230.6 kN	1338.3 kN	1507.0 kN	1604.6 kN	1671.9 kN
Difference $\frac{X-EC}{EC}$	-8.04 %	0.00 %	8.75 %	22.46 %	30.39 %	35.86 %

The deformation when the ultimate load is reached of the various analyses are given in Figure 60 and as can be seen, they follow the buckling shape as obtained by the LBA. The deformations at the ultimate loads confirm that the frame that is considered fails due to an instability problem by 2nd order behavior.

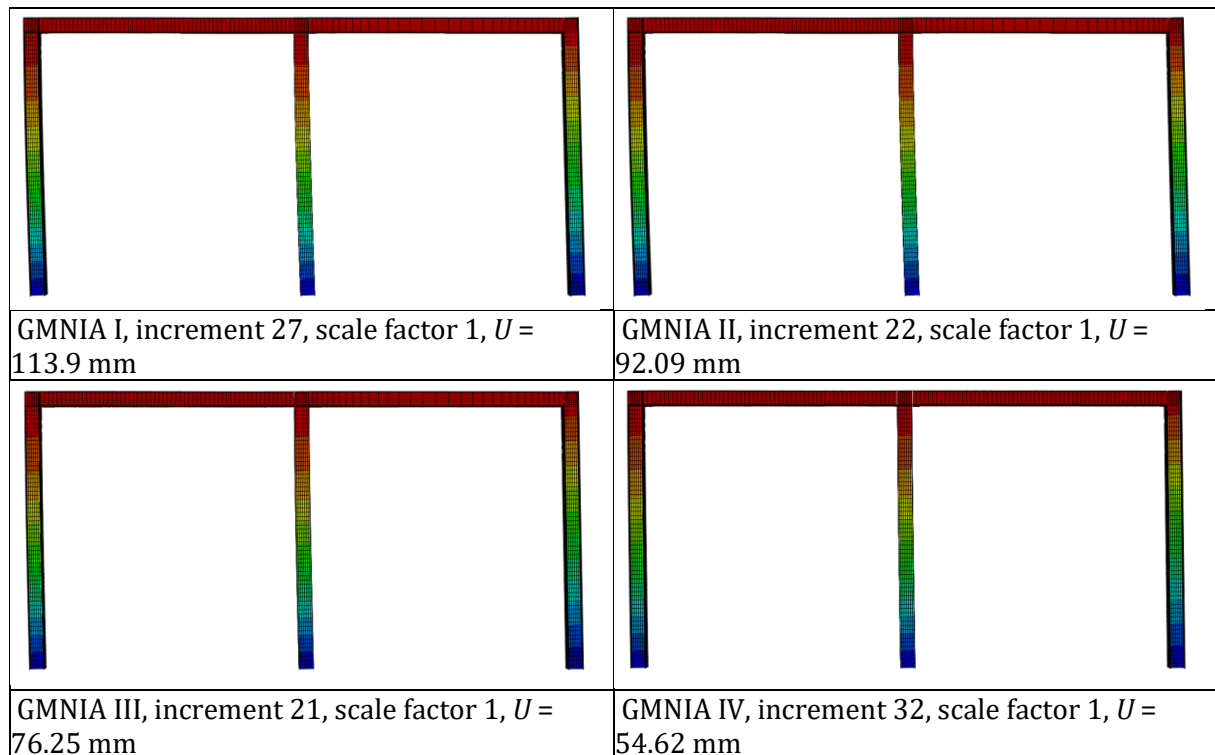


Figure 60: Deformed state of the various analyses when the ultimate load is reached

6.7 INFLUENCE OF THE BUCKLING LENGTH DETERMINATION METHOD

In practice, it is more obvious that a structural engineer will determine L_{cr} differently than creating an extensive FE-model, as done in previous sections. In this paragraph, the influence of different ways of determining the buckling length is studied. Chapter 1 in appendix A gives more background information about the considered options.

Also, since in the previous sections the buckling length from the LBA is used, it means that in the case of the NEN 6771, the leaning column effect is taken into account twice, as already described in 6.6.1. In this paragraph, only combinations of inputs will be used that will lead to only taking into account the leaning column effect once (or not at all in case of the EC in combination with the nomogram). This will also be mentioned in the upcoming sections.

6.7.1.1 L_{cr} according to the nomogram in the Dutch National Annex of the EN 1993-1-1

A common way to determine L_{cr} for a frame is the use of the nomogram from the Dutch National Annex of the EN 1993-1-1 (NB.NA). The nomogram gives the result as the effective length factor $\beta_{nomogram} = L_{cr}/L = \pi/\lambda_n$, where λ_n can be derived with equation (51):

$$C_A \cdot C_B \cdot \lambda_n^2 \cdot \sin(\lambda_n) = (C_A + C_B) \cdot \lambda_n \cdot \cos(\lambda_n) + \sin(\lambda_n), \text{ for } 0 \leq \lambda_n \leq \pi \quad (51)$$

Where A and B refer to the ends of the column and where:

$$C_i = \frac{\sum \frac{I_{cln}}{L_{cln}}}{\sum \mu \frac{I_{bm}}{L_{bm}}} \quad (52)$$

The equation is solved in the following way:

$C_A = \infty$, since the bottom of the frame, is pinned

$\mu = 6$, since the other end of the beam, is rigidly connected to one or more columns which are also loaded in compression.

$I_{cln} = I_{bm} = 1728.5 \cdot 10^5 \text{ mm}^4$ according to 6.4.2

L_{cln} and L_{bm} are both 5000 mm according to Figure 47.

$$C_B = \frac{\sum \frac{I_{cln}}{L_{cln}}}{\sum \mu \frac{I_{bm}}{L_{bm}}} = \frac{\frac{1728.5 \cdot 10^5}{5000}}{6 \cdot \frac{1728.5 \cdot 10^5}{5000}} = 0.167$$

$\infty \cdot 0.167 \cdot \lambda_n^2 \cdot \sin(\lambda_n) = (\infty + 0.167) \cdot \lambda_n \cdot \cos(\lambda_n) + \sin(\lambda_n)$, solving gives $\lambda_n = 1.35$

$$\beta_{nomogram} = \frac{\pi}{\lambda_n} = 2.3279$$

$$L_{cr} = 2.3279 \cdot 5000 = 11639 \text{ mm}$$

When we carry out again the equations for the ultimate resistance load as given in section 6.5 with $L_{cr} = 11639 \text{ mm}$ as input, the following ultimate resistance loads are obtained:

$$N_{ult,DR,NEN,nomogram} = 1447.6 \text{ kN}$$

$$N_{ult,DR,EC,nomogram} = 1580.6 \text{ kN}$$

$$N_{ult,DR,modEC,nomogram} = 1447.6 \text{ kN}$$

Please note that in the case of $N_{ult,DR,EC,nomogram}$ no leaning column effect is considered, and that the results are the magnitude of the ultimate resistance load that will be found in practice by engineers. The new results are plotted in the load-displacement graph, shown in Figure 61.

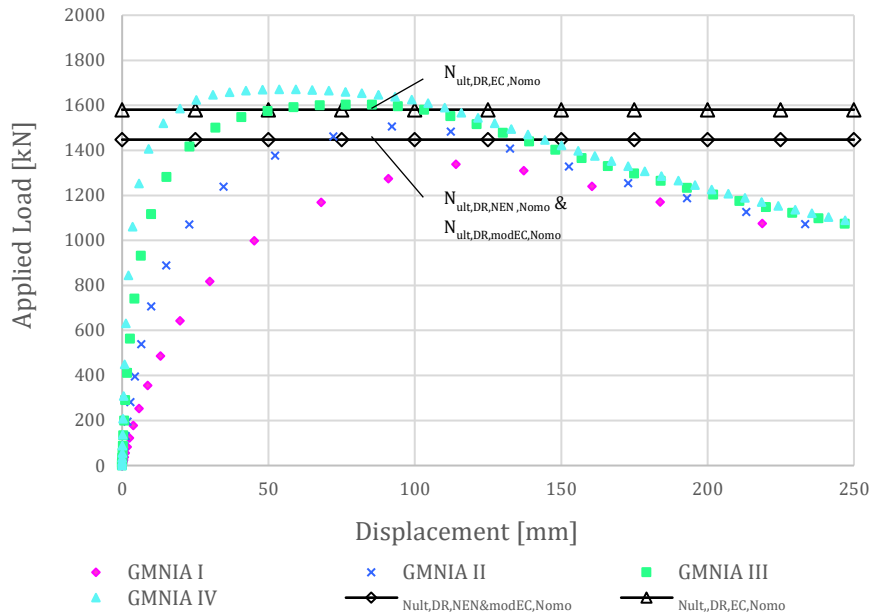


Figure 61: The load-displacement paths for all analyses including the ultimate resistance, where L_{cr} is determined with the nomogram

The result of 11639 mm (nomogram) can be described by the fact that the leaning column effect is not included in the nomogram. When an LBA is performed for a frame without a leaning column, a load factor α_{cr} of 2.6010 is found. This value and the corresponding buckling shape is shown in Figure 62. When this α_{cr} is used in the Euler buckling equation, the following L_{cr} is found:

$$L_{cr} = \pi \cdot \sqrt{\frac{E \cdot I}{\alpha_{cr} \cdot N_{Ed}}} = \pi \cdot \sqrt{\frac{210000 \cdot 1.7285 \cdot 10^8}{2.6010 \cdot 1000 \cdot 10^3}} = 11736 \text{ mm}$$

When we carry out again the design rules for the ultimate resistance loads as given in section 6.5 with $L_{cr} = 11736 \text{ mm}$ as input, results are found that only differ +/- 1 % from the results obtained with the nomogram. Therefore it is stated that these results are the same.

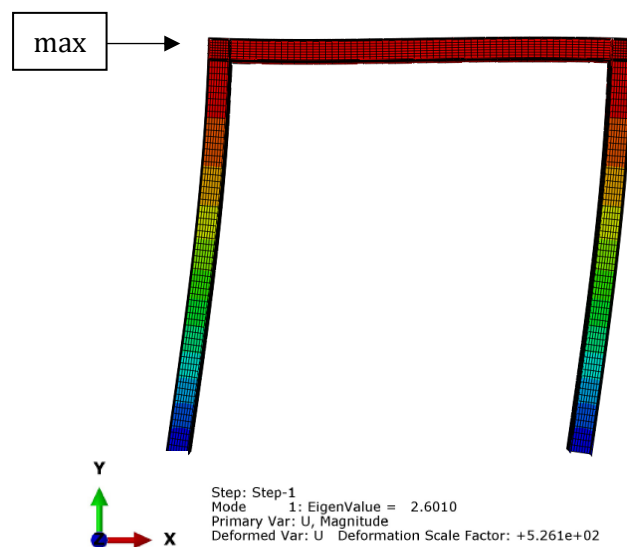


Figure 62: LBA without leaning column with a corresponding $\alpha_{cr} = 2.6010$

6.7.1.2 L_{cr} according to extended American nomogram by L.F. Geschwindner [1]

The American codes for structural safety, LRFD, and AISC [16] specifications make also use of a nomogram to determine the buckling length. The nomogram results in a factor K that has to be multiplied with the height of the column, which is the same procedure as the Eurocode with β . The American equation is given in equation (53):

$$F_{cr} = \frac{(\pi^2 EI)}{(K_{exact} \cdot L)^2} \quad (53)$$

Here can be seen that equation (53) is the same equation as (4), and that gives that $K_{exact} = \beta$. The value of K_{exact} can be derived with the following equation:

$$\frac{G_A G_B (\pi/K)^2 - 36}{6(G_A + G_B)} = \frac{\pi/K}{\tan(\pi/K)} \quad (54)$$

Where A and B refer to the ends of the column and where:

$$G_i = \frac{\sum I_{cln}/L_{cln}}{\sum I_{bm}/L_{bm}} \quad (55)$$

The nomogram equation without the leaning column effect is solved in the following way:

$G_A = \infty$, since the bottom of the frame is pinned

$I_{cln} = I_{bm} = 1728.5 \cdot 10^5 \text{ mm}^4$ according to 6.4.2

L_{cln} and L_{bm} are both 5000 mm according to Figure 47.

$$G_B = \frac{\sum \frac{I_{cln}}{L_{cln}}}{\sum \frac{I_{bm}}{L_{bm}}} = \frac{\frac{1728.5 \cdot 10^5}{5000}}{\frac{1728.5 \cdot 10^5}{5000}} = 1$$

$$\frac{\infty \cdot 1 \cdot (\pi/K)^2 - 36}{6 \cdot (\infty + 1)} = \frac{\pi/K}{\tan(\pi/K)}, \text{ solving gives } K = 2.3279$$

$$L_{cr} = 2.3279 \cdot 5000 = 11639 \text{ mm}$$

The result that is obtained, namely $L_{cr} = 11639 \text{ mm}$, is the same result as the nomogram from the Dutch National Annex (section 6.7.1.1). However, American researcher L.F. Geschwindner [1] extended equation (54) to include the leaning column effect. Any steps for extending equation (54) are unknown, but Geschwindner [1] came up with the following equation to include the leaning column in the American nomogram:

$$\frac{G_A G_B (\pi/K)^2 - 36}{6(G_A + G_B)} \left(1 + \frac{Q}{F}\right) - \frac{\pi/K}{\tan(\pi/K)} \left(1 + \frac{Q}{F}\right) + \frac{6 \tan\left(\frac{\pi}{2K}\right)}{(G_A + G_B) \left(\frac{\pi}{2K}\right)} \left(\frac{Q}{F}\right) + \left(\frac{Q}{F}\right) = 0 \quad (56)$$

The extended nomogram equation is solved in the following way:

$G_A = \infty$, according to the above

$$G_B = \frac{\sum \frac{I_{cln}}{L_{cln}}}{\sum \frac{I_{bm}}{L_{bm}}} = \frac{\frac{1728.5 \cdot 10^5}{5000}}{2 \cdot \frac{1728.5 \cdot 10^5}{5000}} = 0.5, \text{ since now the leaning column is included}$$

$$\frac{\infty \cdot 0.5 \cdot (\pi/K)^2 - 36}{6 \cdot (\infty + 0.5)} \cdot \left(1 + \frac{1}{2}\right) - \frac{\pi/K}{\tan(\pi/K)} \cdot \left(1 + \frac{1}{2}\right) + \frac{6 \tan\left(\frac{\pi}{2K}\right)}{(\infty + 0.5) \left(\frac{\pi}{2K}\right)} \cdot \frac{1}{2} + \frac{1}{2} = 0$$

Solving the above gives the effective length factor $K_{exact} = 2.59$, which gives $L_{cr} = 2.59 \cdot 5000 = 12971$ mm. When we carry out again the equations for the ultimate resistance load as given in section 6.5 with $L_{cr} = 12971$ mm as input, the following ultimate resistance load is obtained:

$$N_{ult,DR,EC,L.F.G} = 1383.8 \text{ kN}$$

In this case, only the EN 1993-1-1 design rule is considered since the leaning column effect is now already taken into account in L_{cr} . The new ultimate resistance load is shown in Figure 63.

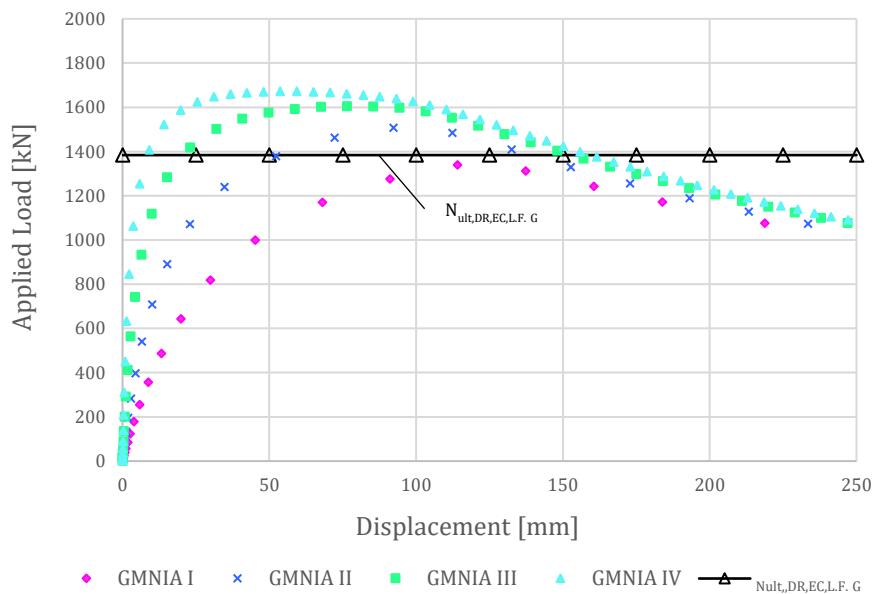


Figure 63: Ultimate resistance load where L_{cr} is determined with the equation from Geschwindner

6.7.1.3 L_{cr} according to the Yura approach (equilibrium)

J.A. Yura invented a method to include the leaning column effect in the buckling length. The approach makes use of simple equilibrium combined with the Euler buckling equation. Consider the leaning column situation in a deformed frame as shown in Figure 64. For equilibrium of the leaning column CD, load Q must be balanced with a horizontal force with a magnitude of $Q\Delta/L$. When this horizontal force is applied in point B to reach equilibrium, one can see that in the deformed state the bending moment in point A will increase with $L \cdot (Q \cdot \Delta) / L$, resulting in a bending moment of $\Delta F + \Delta Q$. This indicates that column AB is loaded by $F + Q$. The solutions for the leaning column load F_{tot} in the NEN 6771 are determined in the same way. However, where the NEN 6771 used this load in the load side of the design rule, Yura used the equilibrium approach to rewrite the Euler buckling equation. This has led to a factor based on the ratio between the sum of the load and the applied load on the stabilizing frame. [1]

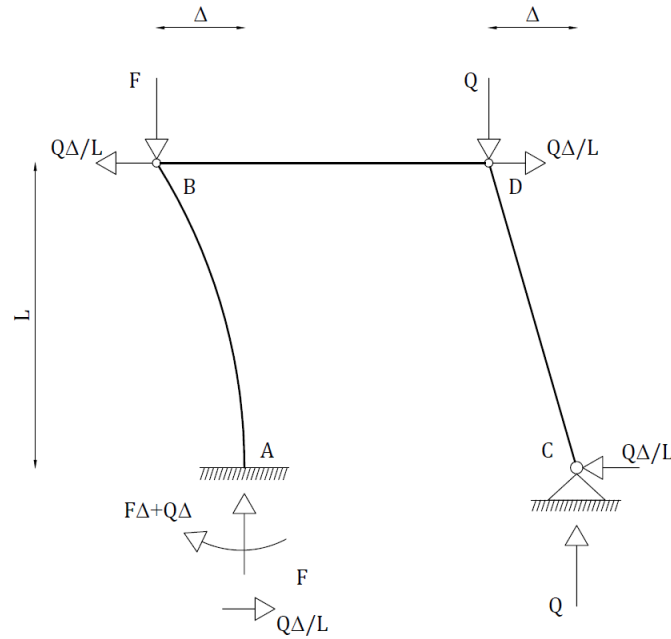


Figure 64: Equilibrium forces in stabilizing column (A-B) and leaning column (C-D) for Yura approach

To derive this factor, we will start with the American notation of the Euler buckling solution for a case with a leaning column load, given in equation (57):

$$F + Q = \frac{\pi^2 EI}{K_o^2 L^2} \quad (57)$$

Where K_o is the modified effective length factor which does not account for the leaning column. To include the leaning column effect in the effective length factor, equation (57) can be rewritten as the following:

$$F = \frac{\pi^2 EI}{K_n^2 L^2} \quad (58)$$

Where K_n in equation (58) is the modified effective length factor which does account for the leaning column effect. Loads F and Q in equation (57) and (58) needs to be replaced with the sum of loads F and Q (ΣF and ΣQ) in case of a frame with more than one leaning column and more than one restraining column. According to Geschwindner [1], it should be noted that this approach maintains the assumption that all restraining columns in a story buckle in a sway mode simultaneously. Solving $\frac{K_n^2}{K_o^2} = \frac{\Sigma F + \Sigma Q}{\Sigma F}$ gives:

$$K_n = K_o \sqrt{\frac{\Sigma F + \Sigma Q}{\Sigma F}} \quad (59)$$

Factors K_o and K_n are American notations for the modified effective length factor, where K_n takes into account the leaning column load, and K_o does not. In the Eurocode the notation for the modified effective length factor is β , and in the case of the nomogram, it does not take into account the leaning column load. This is already experienced in section 6.7.1.1. and for that reason, we now call $K_o = \beta_{nomogram}$, where $\beta_{nomogram}$ is the modified effective length factor obtained from section 6.7.1.1. Substituting the above in (59) gives:

$$\beta_{Yura} = \beta_{nomogram} \cdot \sqrt{\frac{\Sigma F + \Sigma Q}{\Sigma F}} \quad (60)$$

Applying equation (60) for the frame gives:

$\beta_{nomogram} = 2.3279$, according to section 6.7.1.1.

$$\frac{\Sigma F + \Sigma Q}{\Sigma F} = \frac{2 + 1}{2} = \frac{3}{2}$$

$$\beta_{Yura} = 2.3279 \cdot \sqrt{\frac{3}{2}} = 2.85$$

Solving the above gives the effective length factor $\beta_{Yura} = 2.85$, gives $L_{cr} = 2.85 \cdot 5000 = 14255$ mm. When we carry out again the equations for the ultimate resistance load as given in section 6.5 with $L_{cr} = 14255$ mm as input, the following ultimate resistance loads are obtained:

$$N_{ult,DR,EC,Yura} = 1217.6 \text{ kN}$$

Once again only the EN 1993-1-1 design rule is considered since the leaning column effect is already taken into account in L_{cr} . The new ultimate resistance load is shown in Figure 65.

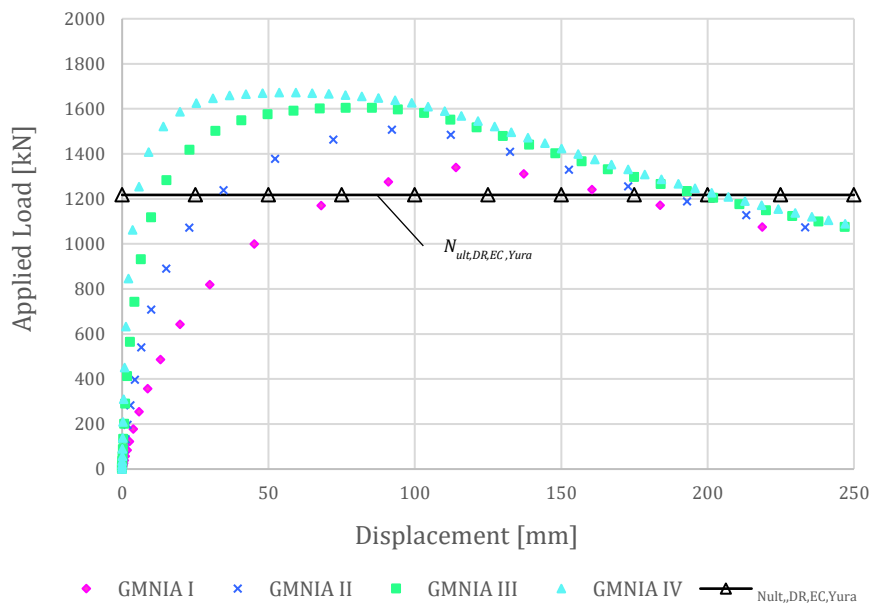


Figure 65: Ultimate resistance load where L_{cr} is determined with the Yura approach

6.7.1.4 Comparison

In section 6.7.1.1, the ultimate resistance loads are determined which in practice would be found by the engineers. It can be seen that $N_{ult,DR,i,nomogram}$ are way higher than $N_{ult,DR,i,LBA}$. These higher results lead to less conservative results. Table 12 shows that the difference for $N_{ult,DR,NEN,nomogram}$ and $N_{ult,DR,EC,nomogram}$ are 27.93% and 28.44% respectively compared to the results obtained with the LBA. If we compare $N_{ult,DR,NEN,nomogram}$ to the various analyses, it still covers the ultimate loads of GMNIA II, III, and IV, as can be seen in Table 13, where the positive percentages indicate that the design-resistance load covers that type of analysis. Section 6.6.1 explained that the results of GMNIA III and GMNIA IV are assumed as closest to reality and that GMNIA I and GMNIA II give

somewhat conservative results. This means that the results of the NEN 6771 are safe due to the bigger margin compared to GMNIA III and GMNIA IV. When we compare $N_{ult,DR,EC,nomogram}$ to the various analyses, the result is almost equal to GMNIA III and GMNIA IV, namely 1.52% and 5.78% respectively (Table 13). This indicates that the results of the EN 1993-1-1 for this frame is safe, but with a small margin.

When the extended American nomogram equation of Geschwindner [1] is considered (section 6.7.1.2) it results in a lower, and more realistic, ultimate resistance load compared to $N_{ult,DR,EC,nomogram}$. This leads to more conservative results, where $N_{ult,DR,EC,L.F.G}$ has a margin of 15.96% and 20.82% from GMNIA III and GMNIA IV respectively (Table 13). But when we compare the results to the most realistic one, $N_{ult,DR,EC,L.F.G}$ still differ 12.45% from $N_{ult,DR,EC,LBA}$ (Table 12).

The Yura approach makes use of the same equilibrium consideration as the solutions for F_{tot} in the NEN 6771. But instead of using the equilibrium approach in the load side of the equation, Yura used it in the derivations of L_{cr} . This different approach leads to different but more accurate results (6.7.1.3). With the use of the Yura approach, $N_{ult,DR,EC,Yura}$ has a margin of 31.78% and 37.31% from GMNIA III and GMNIA IV respectively (Table 13), which is almost the same as the results obtained with the LBA. The high accuracy of the Yura approach is also confirmed by the fact that $N_{ult,DR,EC,Yura}$ only differ -1.06% from $N_{ult,DR,EC,LBA}$ (Table 12).

Since the ultimate resistance loads of the NEN 6771 and the modified EN 1993-1-1 takes into account the leaning column effect twice, the result is considered useless. For that reason, this result in Table 12 has a strikethrough.

Table 12: Results of the various ultimate resistance loads and compared to the most realistic input, where the strikethrough indicates a useless result due to taking the leaning column effect into account twice

	Ultimate DR load (N_{ult}) in kN		Error compared to LBA in %	
	$N_{ult,DR,NEN}$ & $N_{ult,DR,modEC}$	$N_{ult,DR,EC}$	% $N_{ult,DR,NEN}$ & $N_{ult,DR,modEC}$	% $N_{ult,DR,EC}$
LBA with leaning column	1131.6 kN	1230.6 kN	-	0.00 %
Nomogram	1446.6 kN	1580.6 kN	27.93 %	28.44 %
Geschwindner	-	1383.8 kN	-	12.45 %
Yura	-	1217.6 kN	-	-1.06 %

Table 13: Differences between the ultimate loads (horizontal) and the ultimate resistance loads (vertical)

Difference $\frac{F_{ult}-N_{ult}}{N_{ult}}$	Ultimate DR load (N_{ult}) in kN	Ultimate load (F_{ult}) in kN			
		$F_{ult,GMNIA I}$	$F_{ult,GMNIA II}$	$F_{ult,GMNIA III}$	$F_{ult,GMNIA IV}$
		1338.3 kN	1507.0 kN	1604.6 kN	1671.9 kN
$N_{ult,DR,EC,LBA}$	1230.6 kN	8.75%	22.46%	30.39%	35.86%
$N_{ult,DR,NEN,nomogram}$ & $N_{ult,DR,modEC,nomogram}$	1446.6 kN	-7.55%	4.10%	10.85%	15.48%
$N_{ult,DR,EC,nomogram}$	1580.6 kN	-15.33%	-4.66%	1.52%	5.78%
$N_{ult,DR,EC,L.F.G}$	1383.8 kN	-3.29%	8.90%	15.96%	20.82%
$N_{ult,DR,EC,Yura}$	1217.6 kN	9.91%	23.77%	31.78%	37.31%

6.8 CONCLUSIONS

GMNIA III and GMNIA IV were on the forehand assumed to reflect reality in the best way due to realistic modeling of imperfection shapes and realistic residual stresses. The difference between GMNIA III and GMNIA IV is confirmed by the statement of Shayan et al. [9], which is that the use of only the first critical eigenmode (which is the case for GMNIA III) leads to conservative results. The results of GMNIA I and GMNIA II are conservative due to the equivalent imperfections that are present in the design rules for imperfections of the EN 1993-1-1 Clause 5.3.2(3) and 5.3.2(11). Due to the above, GMNIA IV will be used as the ultimate load that the frame can carry.

The modified EC1991-1-1 design rule gives the same result as the NEN 6771, as intended. The combination of the NEN 6771 (and modified EN 1993-1-1) and the nomogram is not considered anymore since this takes into account the leaning column effect twice. With the most realistic way of determining L_{cr} , the LBA with the leaning column, the ultimate resistance load from the current EN 1993-1-1 cover the ultimate load obtained with GMNIA IV with a high conservatism margin (35.9%). This means that the leaning column effect is taken into account in the LBA. However, in practice, a more common way of determining L_{cr} is used, namely the nomogram in the Dutch National Annex of the EN 1993-1-1, where the leaning column effect is not present in the buckling length (equation (49)). When the nomogram is combined with the NEN 6771 (and modified EN 1993-1-1) and the current EN 1993-1-1 design rule, higher ultimate resistance loads are found, roughly 28% higher than the LBA. When these ultimate resistance loads are compared to the ultimate load obtained with GMNIA IV, conservatism margins of 15.48% (NEN&modEN) and 5.78% (current EN) are found. This means that the design rules combined with the nomogram lead to safe results, but only with a small conservatism margin, where the margins decreased almost 20% compared to the LBA.

The decrease of 20% suggests that there can be a lot gained with including the leaning column effect in the buckling length. In the literature, there have been searched for practical methods to include the leaning column in the buckling length. This has led to two methods, the extended American nomogram equation of Geschwindner and the equilibrium approach of Yura. The current EN 1993-1-1 design rule in combination with the extended nomogram equation of Geschwindner lowers the ultimate resistance load but still differs 12.5% with the LBA, meaning that the approach is not accurate. The Yura approach in combination with the EN 1993-1-1, however, gives results that are close to the LBA, with only a difference of -1.06%. The use of the Yura approach leads to a conservatism margin of 37.31%, which is almost the same margin obtained with the LBA.

All these conservatism margins as discussed above can be grouped into three different types and translated into three unity check graphs. The three different types are:

- The results where the leaning column effect is included in the load-side of the design rule equation (NEN 6771 & modified EN 1993-1-1 in combination with the nomogram)
- The results where the leaning column effect is not included (EN 1993-1-1 in combination with the nomogram)
- The results where the leaning column effect is included in the buckling length (EN 1993-1-1 in combination with the LBA, Geschwindner, and Yura)

The unity checks are obtained by dividing the ultimate resistance loads ($N_{ult,i,j}$) with the ultimate load that the frame can carry according to GMNIA IV ($F_{ult,GMNIA IV}$), so are presenting the ratio between N_{ult} and F_{ult} . The unity check graphs are shown in Figure 66 and Figure 67, where the determination of the values is shown in the graph. These graphs visualize the conservatism margin whether they are safe or unsafe, where higher unity checks lead to lower conservatism margins. For the frame considered in this chapter, the margins are all on the safe side.

As already discussed before, the modified EC1991-1-1 design rule gives the same result as the NEN 6771. This means that converting the NEN 6771 into a modified EN 1993-1-1 equation, and therefore a clearer presentation of the leaning column effect, is possible.

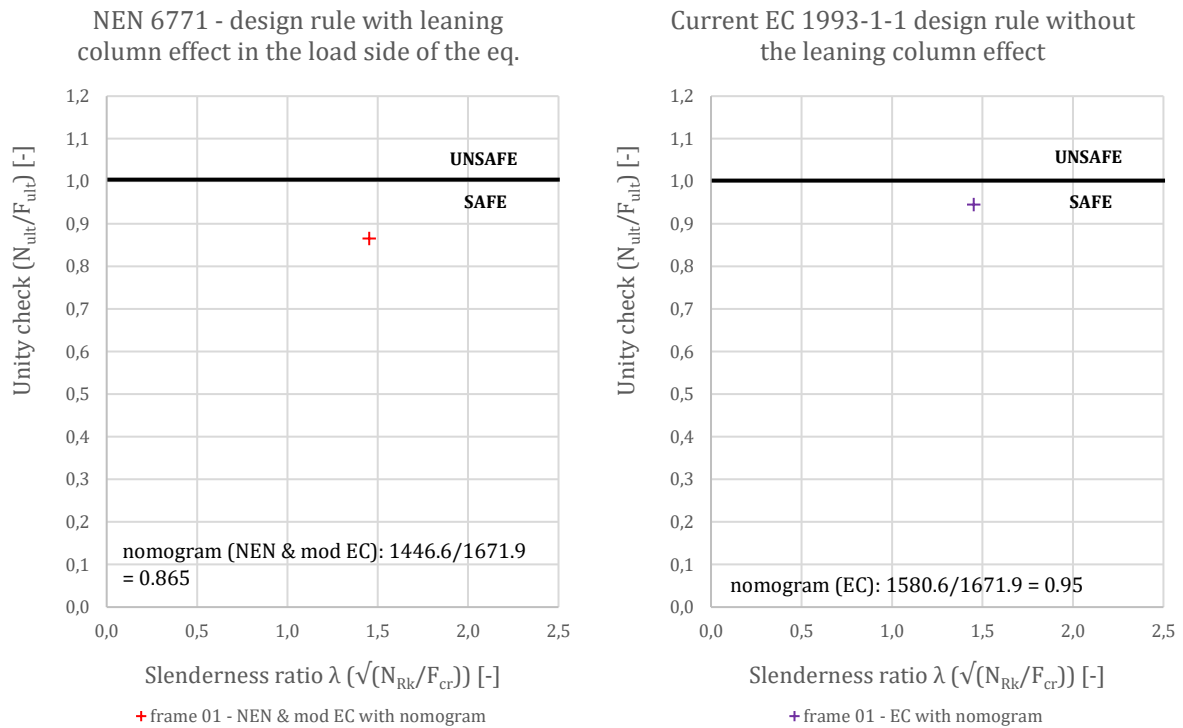


Figure 66: Unity check graphs for the NEN 6771& mod EN 1993-1-1 (left) and current EN 1993-1-1 (right)

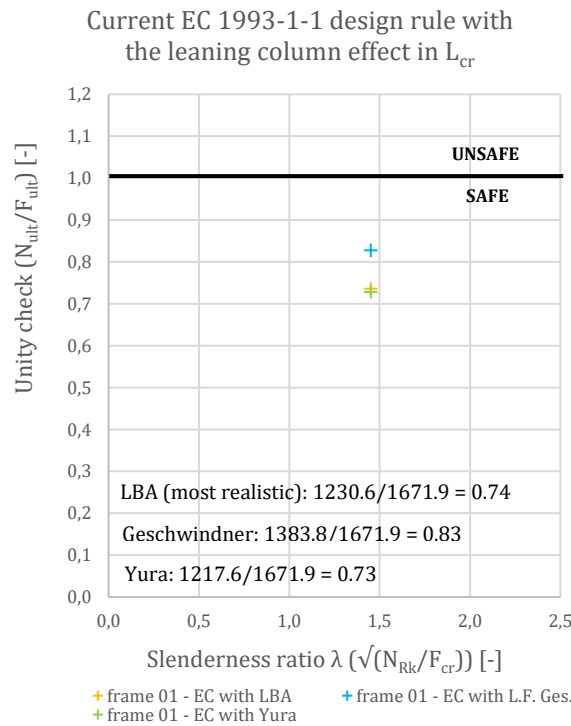


Figure 67: Unity check graph for the EN 1993-1-1 in combination with the LBA and Yura

7 PARAMETRIC STUDY

7.1 INTRODUCTION

In chapter 6, a case study is elaborated for one sway frame that is sensitive to the leaning column effect, loaded with only a small leaning column load and a horizontal load due to imperfections of the columns. In this chapter, the frame configuration will be analyzed with different dimensions. These parameters will be discussed in section 7.2. This is to study the leaning column effect for a range of simple cases that are the most vulnerable for the leaning column effect, but only with small leaning column and horizontal loads. The frame configurations as already described in the research scope (section 1.4) will also be studied. Each frame configuration will be analyzed fifteen times. These different parameters and frame configurations aim to cover a large range of leaning column problems. In the upcoming sections, a more in-depth description will be given for each frame configuration and the corresponding parameters. The parametric study has yet to prove whether all those margins as discussed in section 6.8 are safe for all cases.

In sections 7.2 and 7.3 the various parameters and frame configurations will be discussed. The input for and from these sections is given in appendix B. The most important inputs from appendix B will be given in table format per frame. In these tables, the abbreviations will be used as already used in the previous chapters. Value Δ is the imperfection magnitude to scale the normalized matrix of displacements of the first out-of-plane eigenmode in GMNIA III, values A_1 , A_2 , and A_3 are the imperfection magnitudes to scale the normalized matrix of displacements of the first three eigenmodes for GMNIA IV. For GMNIA IV, up to three eigenmodes will be used, as discussed in section 3.3.4. However, section 6.4.5 already showed that not for all cases three side-sway eigenmodes will be found, while only side-sway eigenmodes are desired to obtain the ultimate load that the frame can carry. In the upcoming frame configurations a variation of 1, 2, or 3 critical eigenmodes for GMNIA IV are used, depending on the stiffness of the total frame. When $A_3 = 0$ or $A_3 = A_2 = 0$ it indicates that only 2 or 1 eigenmodes respectively are used. Section 7.4 shows the results of the parametric study in table format. Behind the results of the ultimate design resistance loads goes a large calculation sheet, which is given in appendix C. The scripts that are used to carry out the parametric study in Abaqus are given in appendix D.

In this chapter, the results with the extended nomogram equation of Geschwindner are not considered anymore, due to the lack of accuracy.

7.2 PARAMETERS

As mentioned before, various frame configurations will be analyzed with various parameters, all aiming to cover a large range of leaning column problems. The various parameters that will be considered are based on different stiffness and slenderness ratios. It is expected that how higher the slenderness and stiffness ratio, how higher the influence of the leaning column effect. The stiffness ratio is defined by $C = \frac{I_{cln}/L_{cln}}{I_{bm}/L_{bm}}$, where I_{cln} and I_{bm} are the second moment of inertia of the column and beam respectively, and where L_{cln} and L_{bm} are the lengths of the column and beam respectively. The stiffness ratio will be pinned on values 1, 1.5, and 2. In ratio C only L_{bm} will be variable, meaning that for a frame with the same cross-section for column and beam, the length times x1.5 and x2 already lead to stiffness ratios 1.5 and 2. However, theoretically, this ratio difference can also indicate different cross-sections (I_i) between column and beam, which can lead to less stiff structures.

The slenderness ratio is defined by $\bar{\lambda} = \sqrt{\frac{N_{Rk}}{F_{cr}}}$, where $F_{cr} = \frac{\pi^2 EI}{L_{cr}^2}$ as given in equation (4) and $N_{Rk} = A \cdot f_y$. For obtaining different slenderness ratios, the frame is performed with a different cross-section of the column. This gives that the cross-section properties in the equations will be variable, which are A and EI . The smaller the cross-section over the height, the bigger the slenderness ratio. Please note that the same cross-section is used for the beam.

The L_{cr} of a structure depends on the stiffness of a structure, so depends on ratio C . This means that the frames with a different C also have a slightly different $\bar{\lambda}$. For determining $\bar{\lambda}$, the most realistic parameters will be used, and that is the α_{cr} from the LBA. Value α_{cr} from the LBA includes the leaning column effect and this leads to a higher slenderness ratio. These higher slenderness ratios will be used to plot all the results. In practice these slenderness ratios $\bar{\lambda}$ will not be found due to the lack of the leaning column effect in the nomogram.

The frame that is elaborated in chapter 6 is again considered in the parametric study and performed with 1.5x and 2x the width. The other twelve frames are based on a height and width of 3000 mm. The corresponding cross-sections and ratios are given in Table 14.

Table 14: Visualization of the frames that are obtained with the variable parameters

	$b=5000, C=1$	$b=7500, C=1.5$	$b=10000, C=2$
HEA300 $\bar{\lambda}$ varies			
	$b=3000, C=1$	$b=4500, C=1.5$	$b=6000, C=2$
HEA120 $\bar{\lambda}$ varies			
HEA160 $\bar{\lambda}$ varies			
HEA220 $\bar{\lambda}$ varies			
HEA280 $\bar{\lambda}$ varies			

7.3 FRAME CONFIGURATION

7.3.1 Frame A

The first frame configuration that is elaborated is already explained in section 6.1 and shortly introduced in section 7.1. It is a sway frame that is sensitive to the leaning column effect, loaded with a vertical leaning column load $Q = F$, and a horizontal load of $2\phi F + \phi Q$, as shown in Figure 68, where ϕ (equation 44) is the slope of the structure due to imperfections.

Table 15 gives the corresponding parameters and input-values when combining the various fifteen frames from section 7.2 with frame configuration A. The parameters and input-values in the table are the most interesting regarding F_{ult} and N_{ult} . The determination of the values for frame

01 is already explained in detail in sections 6.4 and 6.5. The slenderness ratio varies from 0.96 to 2.56. For all corresponding values see appendix B.

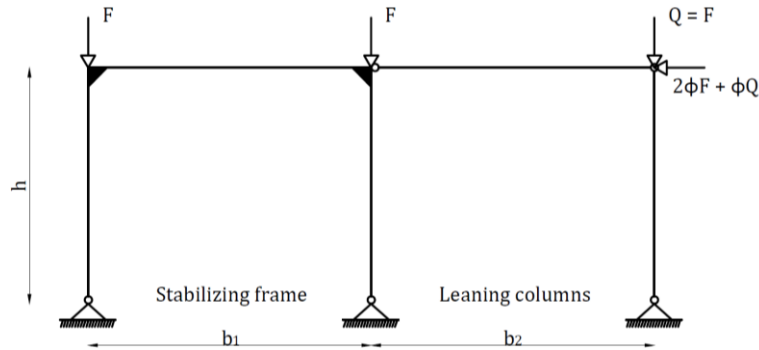


Figure 68: Frame A – Pin-ended moment-resisting frame with a hinged leaning column

Table 15: Input values for frame A to carry out GMNIA III and GMNIA IV

H-section	h	b ₁ =b ₂	F	Q	H	φ	λ	C	α _{cr}	L _{cr,LBA}	L _{cr,nomogram}	L _{cr,Yura}	Δ	A ₁	A ₂	A ₃	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame 01	HEA300	5000	5000	1000	1000	0	3.65E-03	1.45	1	1.79	14148	11639	14255	12.93	-6.12	1.71	0
frame 02	HEA300	5000	7500	1000	1000	0	3.65E-03	1.55	1.5	1.58	15073	12421	15213	13.37	-6.12	1.71	0
frame 03	HEA300	5000	10000	1000	1000	0	3.65E-03	1.64	2	1.40	15991	13173	16133	13.75	-6.12	1.71	0
frame A1	HEA120	3000	3000	1000	1000	0	4.08E-03	2.25	1	0.17	8432	6984	8553	7.73	-3.44	0.89	1.19
frame A2	HEA120	3000	4500	1000	1000	0	4.08E-03	2.41	1.5	0.15	9014	7453	9128	8.01	-3.44	0.89	1.19
frame A3	HEA120	3000	6000	1000	1000	0	4.08E-03	2.56	2	0.13	9576	7904	9680	8.24	-3.44	0.89	1.19
frame A4	HEA160	3000	3000	1000	1000	0	4.08E-03	1.68	1	0.46	8458	6984	8553	7.74	-3.67	1.02	0
frame A5	HEA160	3000	4500	1000	1000	0	4.08E-03	1.80	1.5	0.41	9027	7453	9128	8.01	-3.67	1.02	0
frame A6	HEA160	3000	6000	1000	1000	0	4.08E-03	1.91	2	0.36	9583	7904	9680	8.24	-3.67	1.02	0
frame A7	HEA220	3000	3000	1000	1000	0	4.08E-03	1.22	1	1.47	8546	6984	8553	7.79	-3.68	0	0
frame A8	HEA220	3000	4500	1000	1000	0	4.08E-03	1.30	1.5	1.30	9083	7453	9128	8.04	-3.68	0	0
frame A9	HEA220	3000	6000	1000	1000	0	4.08E-03	1.37	2	1.16	9624	7904	9680	8.26	-3.68	0	0
frame A10	HEA280	3000	3000	1000	1000	0	4.08E-03	0.96	1	3.58	8673	6984	8553	7.85	-3.68	0	0
frame A11	HEA280	3000	4500	1000	1000	0	4.08E-03	1.01	1.5	3.21	9167	7453	9128	8.07	-3.68	0	0
frame A12	HEA280	3000	6000	1000	1000	0	4.08E-03	1.07	2	2.87	9687	7904	9680	8.28	-3.68	0	0

7.3.2 Frame A-NL

Frame A-NL is a sway frame without the leaning column. This frame configuration is performed to see whether the conservatism margin between the design rules and GMNIA IV is comparable to the other frame configurations, so for validation reasons. This means that the leaning column load is reduced to 0, so $Q = 0$, leading to a horizontal load of $2\phi F$, as shown in Figure 69. The most interesting inputs for frame A-NL are given in Table 16. The slenderness ratio varies from 0.79 to 2.11 and differs slightly from frame A due to the absence of the leaning column effect. For all corresponding values see appendix B.

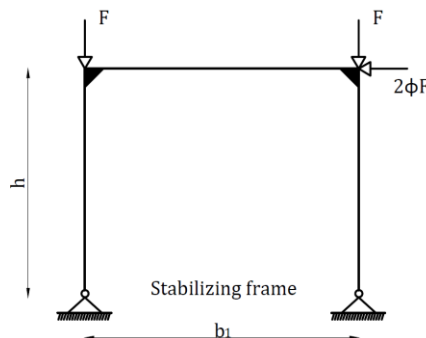


Figure 69: Frame A-NL – pin-ended moment-resisting frame without a leaning column

Table 16: Input values for frame A-NL to carry out GMNIA III and GMNIA IV

H-section	h	b ₁ =b ₂	F	Q	H	φ	λ	C	α _{cr}	L _{cr,LBA}	L _{cr,nomogram}	L _{cr,Yura}	Δ	A ₁	A ₂	A ₃	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame 01-NL	HEA300	5000	5000	1000	0	0	3.65E-03	1.20	1	2.60	11736	11639	11639	11.48	5.73	1.48	1.99
frame 02-NL	HEA300	5000	7500	1000	0	0	3.65E-03	1.28	1.5	2.31	12456	12421	12421	11.97	5.73	1.48	1.99
frame 03-NL	HEA300	5000	10000	1000	0	0	3.65E-03	1.35	2	2.06	13178	13173	13173	12.41	5.73	1.48	1.99
frame A1-NL	HEA120	3000	3000	1000	0	0	4.08E-03	1.87	1	0.25	6996	6984	6984	6.85	3.44	0.89	1.19
frame A2-NL	HEA120	3000	4500	1000	0	0	4.08E-03	1.99	1.5	0.22	7449	7453	7453	7.17	3.44	0.89	1.19
frame A3-NL	HEA120	3000	6000	1000	0	0	4.08E-03	2.11	2	0.19	7892	7904	7904	7.44	3.44	0.89	1.19
frame A4-NL	HEA160	3000	3000	1000	0	0	4.08E-03	1.40	1	0.67	7017	6984	6984	6.87	3.44	0.89	1.19
frame A5-NL	HEA160	3000	4500	1000	0	0	4.08E-03	1.48	1.5	0.59	7460	7453	7453	7.17	3.44	0.89	1.19
frame A6-NL	HEA160	3000	6000	1000	0	0	4.08E-03	1.57	2	0.53	7898	7904	7904	7.44	3.44	0.89	1.19
frame A7-NL	HEA220	3000	3000	1000	0	0	4.08E-03	1.01	1	2.14	7087	6984	6984	6.92	3.44	0.89	1.19
frame A8-NL	HEA220	3000	4500	1000	0	0	4.08E-03	1.07	1.5	1.91	7505	7453	7453	7.20	3.44	0.89	1.19
frame A9-NL	HEA220	3000	6000	1000	0	0	4.08E-03	1.13	2	1.71	7930	7904	7904	7.46	3.44	0.89	1.19
frame A10-NL	HEA280	3000	3000	1000	0	0	4.08E-03	0.79	1	5.21	7188	6984	6984	6.99	3.67	1.02	0
frame A11-NL	HEA280	3000	4500	1000	0	0	4.08E-03	0.84	1.5	4.70	7572	7453	7453	7.25	3.67	1.02	0
frame A12-NL	HEA280	3000	6000	1000	0	0	4.08E-03	0.88	2	4.23	7981	7904	7904	7.49	3.67	1.02	0

7.3.3 Frame A-V

Frame A-V is similar to frame A, but now with an increased leaning column load of a factor x10. This is performed to study the same frame as frame A, but now with an increased leaning column load, and therefore also higher slenderness ratio. The frame configuration is shown in Figure 70. The leaning column load is increased to 10F, so $Q = 10F$, where the 10F has to represent a row of ten leaning columns. The horizontal load is again $2\phi F + \phi Q$. To avoid that the buckling of the leaning column is the governing failure mode, the leaning column is strengthened and stiffened with $10f_{y,d}$, and $10EI$. The most interesting inputs for frame A-V are given in Table 17. The slenderness ratio varies from 1.88 to 5.05, which is a large increase compared to frame A, and therefore covers a larger range of leaning column problems. For all corresponding values see appendix B.

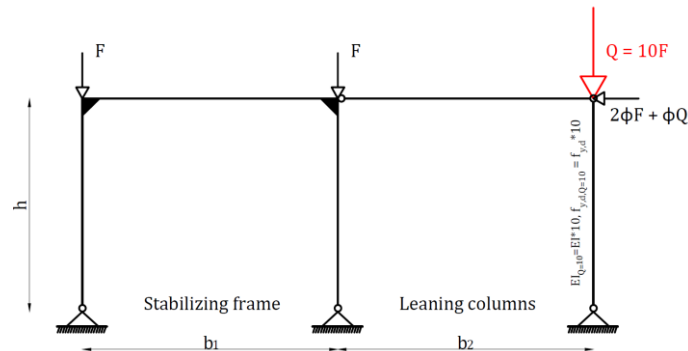


Figure 70: Frame A-V – pin-ended moment-resisting frame with a heavily loaded, stiffer, and stronger, leaning column

Table 17: Input values for frame A-V to carry out GMNIA III and GMNIA IV

H-section	h	$b_1=b_2$	F	Q	H	φ	λ	C	α_{cr}	$L_{cr,LBA}$	$L_{cr,nomogram}$	$L_{cr,Yura}$	Δ	A_1	A_2	A_3	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame 01-V	HEA300	5000	5000	1000	10000	0	3.65E-03	2.84	1	0.47	27681	11639	28511	16.39	6.12	-1.71	0
frame 02-V	HEA300	5000	7500	1000	10000	0	3.65E-03	3.04	1.5	0.41	29663	12421	30426	16.63	6.12	-1.71	0
frame 03-V	HEA300	5000	10000	1000	10000	0	3.65E-03	3.24	2	0.36	31599	13173	32267	16.84	6.12	-1.71	0
frame A1-V	HEA120	3000	3000	1000	10000	0	4.08E-03	4.40	1	0.04	16482	6984	17106	9.82	3.44	-0.89	-1.19
frame A2-V	HEA120	3000	4500	1000	10000	0	4.08E-03	4.73	1.5	0.04	17725	7453	18256	9.97	3.44	-0.89	-1.19
frame A3-V	HEA120	3000	6000	1000	10000	0	4.08E-03	5.05	2	0.03	18908	7904	19360	10.10	3.44	-0.89	-1.19
frame A4-V	HEA160	3000	3000	1000	10000	0	4.08E-03	3.29	1	0.12	16542	6984	17106	9.82	3.67	-1.02	0
frame A5-V	HEA160	3000	4500	1000	10000	0	4.08E-03	3.53	1.5	0.10	17758	7453	18256	9.97	3.67	-1.02	0
frame A6-V	HEA160	3000	6000	1000	10000	0	4.08E-03	3.77	2	0.09	18931	7904	19360	10.10	3.67	-1.02	0
frame A7-V	HEA220	3000	3000	1000	10000	0	4.08E-03	2.39	1	0.38	16734	6984	17106	9.85	3.68	0	0
frame A8-V	HEA220	3000	4500	1000	10000	0	4.08E-03	2.55	1.5	0.34	17885	7453	18256	9.99	3.68	0	0
frame A9-V	HEA220	3000	6000	1000	10000	0	4.08E-03	2.71	2	0.30	19029	7904	19360	10.11	3.68	0	0
frame A10-V	HEA280	3000	3000	1000	10000	0	4.08E-03	1.88	1	0.93	17009	6984	17106	9.88	3.68	0	0
frame A11-V	HEA280	3000	4500	1000	10000	0	4.08E-03	1.99	1.5	0.82	18074	7453	18256	10.01	3.68	0	0
frame A12-V	HEA280	3000	6000	1000	10000	0	4.08E-03	2.12	2	0.73	19179	7904	19360	10.12	3.68	0	0

7.3.4 Frame A-H, A-NLH, and A-VH

Additionally, frame A, frame A-NL, and frame A-V are extended with an additional horizontal load, to see the influence of a larger lateral displacement. This is interesting since, in the case of the design rules, the additional horizontal load only leads to a higher M_{Ed} , but in the case of the GMNIA also to larger lateral displacements, leading to a higher leaning column effect. The horizontal load H is based on a wind load of 1 kN/m^2 , with an area of $h*b$ ($b=b_1=b_2$). This means that the horizontal load is different for some cases since the width is a variable. The additional horizontal loads are shown in red in Figure 71, Figure 72, and Figure 73. The most interesting inputs are given in Table 18, Table 19, and Table 20. Here it can be seen that the input is the same as for frame A, frame A-NL, and Frame A-V, except load H . For all corresponding values see appendix B.

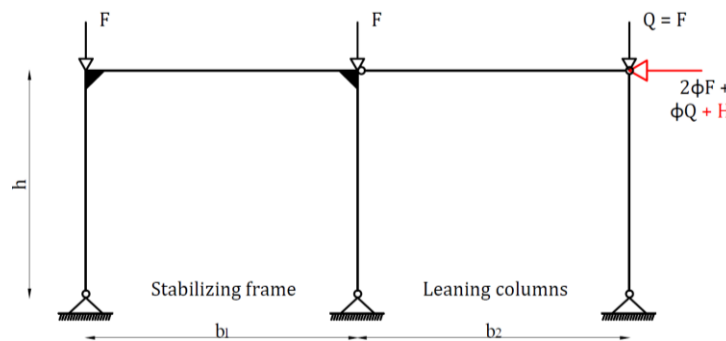


Figure 71: Frame A-H – pin-ended moment-resisting frame with a leaning column and additional horizontal wind load

Table 18: Input values for frame A-H to carry out GMNIA III and GMNIA IV

H-section	h	b ₁ =b ₂	F	Q	H	φ	λ	C	α _{cr}	L _{cr,LBA}	L _{cr,nomogram}	L _{cr,Yura}	Δ	A ₁	A ₂	A ₃	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame 01-H	HEA300	5000	5000	1000	1000	12.5	3.65E-03	1.45	1	1.79	14148	11639	14255	-12.93	-6.12	1.71	0
frame 02-H	HEA300	5000	7500	1000	1000	18.75	3.65E-03	1.55	1.5	1.58	15073	12421	15213	-13.37	-6.12	1.71	0
frame 03-H	HEA300	5000	10000	1000	1000	25	3.65E-03	1.64	2	1.40	15991	13173	16133	-13.75	-6.12	1.71	0
frame A1-H	HEA120	3000	3000	1000	1000	4.5	4.08E-03	2.25	1	0.17	8432	6984	8553	-7.73	-3.44	0.89	-1.19
frame A2-H	HEA120	3000	4500	1000	1000	6.75	4.08E-03	2.41	1.5	0.15	9014	7453	9128	-8.01	-3.44	0.89	-1.19
frame A3-H	HEA120	3000	6000	1000	1000	9	4.08E-03	2.56	2	0.13	9576	7904	9680	-8.24	-3.44	0.89	-1.19
frame A4-H	HEA160	3000	3000	1000	1000	4.5	4.08E-03	1.68	1	0.46	8458	6984	8553	-7.74	-3.67	1.02	0
frame A5-H	HEA160	3000	4500	1000	1000	6.75	4.08E-03	1.80	1.5	0.41	9027	7453	9128	-8.01	-3.67	1.02	0
frame A6-H	HEA160	3000	6000	1000	1000	9	4.08E-03	1.91	2	0.36	9583	7904	9680	-8.24	-3.67	1.02	0
frame A7-H	HEA220	3000	3000	1000	1000	4.5	4.08E-03	1.22	1	1.47	8546	6984	8553	-7.79	-3.68	0	0
frame A8-H	HEA220	3000	4500	1000	1000	6.75	4.08E-03	1.30	1.5	1.30	9083	7453	9128	-8.04	-3.68	0	0
frame A9-H	HEA220	3000	6000	1000	1000	9	4.08E-03	1.37	2	1.16	9624	7904	9680	-8.26	-3.68	0	0
frame A10-H	HEA280	3000	3000	1000	1000	4.5	4.08E-03	0.96	1	3.58	8673	6984	8553	-7.85	-3.68	0	0
frame A11-H	HEA280	3000	4500	1000	1000	6.75	4.08E-03	1.01	1.5	3.21	9167	7453	9128	-8.07	-3.68	0	0
frame A12-H	HEA280	3000	6000	1000	1000	9	4.08E-03	1.07	2	2.87	9687	7904	9680	-8.28	-3.68	0	0

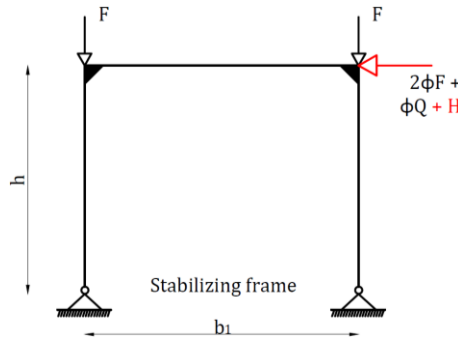


Figure 72: Frame A-NLH: pin-ended moment-resisting frame without a leaning column and with an additional horizontal wind load

Table 19: Input values for frame A-NLH to carry out GMNIA III and GMNIA IV

H-section	h	b ₁ =b ₂	F	Q	H	φ	λ	C	α _{cr}	L _{cr,LBA}	L _{cr,nomogram}	L _{cr,Yura}	Δ	A ₁	A ₂	A ₃	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame 01-NLH	HEA300	5000	5000	1000	0	12.5	3.65E-03	1.20	1	2.60	11736	11639	11639	-11.48	-5.73	-1.48	-1.99
frame 02-NLH	HEA300	5000	7500	1000	0	18.75	3.65E-03	1.28	1.5	2.31	12456	12421	12421	-11.97	-5.73	-1.48	-1.99
frame 03-NLH	HEA300	5000	10000	1000	0	25	3.65E-03	1.35	2	2.06	13178	13173	13173	-12.41	-5.73	-1.48	-1.99
frame A1-NLH	HEA120	3000	3000	1000	0	4.5	4.08E-03	1.87	1	0.25	6996	6984	6984	-6.85	-3.44	-0.89	-1.19
frame A2-NLH	HEA120	3000	4500	1000	0	6.75	4.08E-03	1.99	1.5	0.22	7449	7453	7453	-7.17	-3.44	-0.89	-1.19
frame A3-NLH	HEA120	3000	6000	1000	0	9	4.08E-03	2.11	2	0.19	7892	7904	7904	-7.44	-3.44	-0.89	-1.19
frame A4-NLH	HEA160	3000	3000	1000	0	4.5	4.08E-03	1.40	1	0.67	7017	6984	6984	-6.87	-3.44	-0.89	-1.19
frame A5-NLH	HEA160	3000	4500	1000	0	6.75	4.08E-03	1.48	1.5	0.59	7460	7453	7453	-7.17	-3.44	-0.89	-1.19
frame A6-NLH	HEA160	3000	6000	1000	0	9	4.08E-03	1.57	2	0.53	7898	7904	7904	-7.44	-3.44	-0.89	-1.19
frame A7-NLH	HEA220	3000	3000	1000	0	4.5	4.08E-03	1.01	1	2.14	7087	6984	6984	-6.92	-3.44	-0.89	-1.19
frame A8-NLH	HEA220	3000	4500	1000	0	6.75	4.08E-03	1.07	1.5	1.91	7505	7453	7453	-7.20	-3.44	-0.89	-1.19
frame A9-NLH	HEA220	3000	6000	1000	0	9	4.08E-03	1.13	2	1.71	7930	7904	7904	-7.46	-3.44	-0.89	-1.19
frame A10-NLH	HEA280	3000	3000	1000	0	4.5	4.08E-03	0.79	1	5.21	7188	6984	6984	-6.99	-3.67	-1.02	0
frame A11-NLH	HEA280	3000	4500	1000	0	6.75	4.08E-03	0.84	1.5	4.70	7572	7453	7453	-7.25	-3.67	-1.02	0
frame A12-NLH	HEA280	3000	6000	1000	0	9	4.08E-03	0.88	2	4.23	7981	7904	7904	-7.49	-3.67	-1.02	0

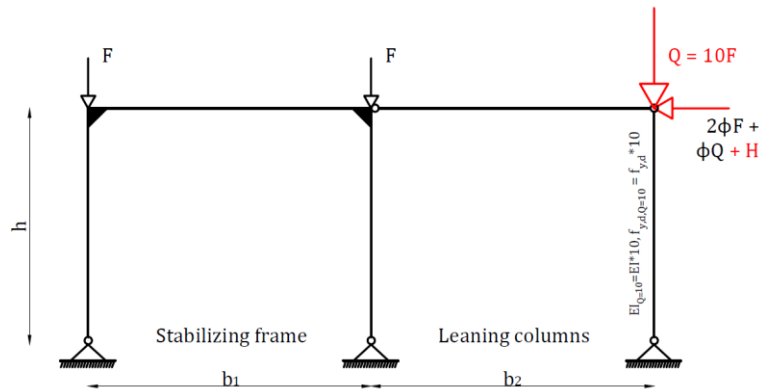


Figure 73: Frame A-VH – pin-ended moment resisting frame with a heavy loaded, stronger and stiffer, leaning column and additional horizontal wind load

Table 20: Input values for frame A-VH to carry out GMNIA III and GMNIA IV

H-section	h	b ₁ =b ₂	F	Q	H	φ	λ	C	α _{cr}	L _{cr,LBA}	L _{cr,nomogram}	L _{cr,Yura}	Δ	A ₁	A ₂	A ₃	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame 01-VH	HEA300	5000	5000	1000	10000	12.5	3.65E-03	2.84	1	0.47	27681	11639	28511	-16.39	-6.12	1.71	0
frame 02-VH	HEA300	5000	7500	1000	10000	18.75	3.65E-03	3.04	1.5	0.41	29663	12421	30426	-16.63	-6.12	1.71	0
frame 03-VH	HEA300	5000	10000	1000	10000	25	3.65E-03	3.24	2	0.36	31599	13173	32267	-16.84	-6.12	1.71	0
frame A1-VH	HEA120	3000	3000	1000	10000	4.5	4.08E-03	4.40	1	0.04	16482	6984	17106	-9.82	-3.44	0.89	-1.19
frame A2-VH	HEA120	3000	4500	1000	10000	6.75	4.08E-03	4.73	1.5	0.04	17725	7453	18256	-9.97	-3.44	0.89	-1.19
frame A3-VH	HEA120	3000	6000	1000	10000	9	4.08E-03	5.05	2	0.03	18908	7904	19360	-10.10	-3.44	0.89	-1.19
frame A4-VH	HEA160	3000	3000	1000	10000	4.5	4.08E-03	3.29	1	0.12	16542	6984	17106	-9.82	-3.67	1.02	0
frame A5-VH	HEA160	3000	4500	1000	10000	6.75	4.08E-03	3.53	1.5	0.10	17758	7453	18256	-9.97	-3.67	1.02	0
frame A6-VH	HEA160	3000	6000	1000	10000	9	4.08E-03	3.77	2	0.09	18931	7904	19360	-10.10	-3.67	1.02	0
frame A7-VH	HEA220	3000	3000	1000	10000	4.5	4.08E-03	2.39	1	0.38	16734	6984	17106	-9.85	-3.68	0	0
frame A8-VH	HEA220	3000	4500	1000	10000	6.75	4.08E-03	2.55	1.5	0.34	17885	7453	18256	-9.99	-3.68	0	0
frame A9-VH	HEA220	3000	6000	1000	10000	9	4.08E-03	2.71	2	0.30	19029	7904	19360	-10.11	-3.68	0	0
frame A10-VH	HEA280	3000	3000	1000	10000	4.5	4.08E-03	1.88	1	0.93	17009	6984	17106	-9.88	-3.68	0	0
frame A11-VH	HEA280	3000	4500	1000	10000	6.75	4.08E-03	1.99	1.5	0.82	18074	7453	18256	-10.01	-3.68	0	0
frame A12-VH	HEA280	3000	6000	1000	10000	9	4.08E-03	2.12	2	0.73	19179	7904	19360	-10.12	-3.68	0	0

7.3.5 Frame B

To check what the leaning column effect does in a more stiff frame, a stiffer mechanical scheme is considered. This frame configuration is called Frame B and is shown in Figure 74, where the bottom ends of the column are clamped and marked in red. For this different mechanical scheme, the buckling length reduces, so a lower slenderness ratio will be obtained. Besides the lower slenderness ratio, this configuration did not lead to any interesting results. Therefore frame B is only performed with a small leaning column load. The parameters and input-values are given in Table 21. For all corresponding values see appendix B. The slenderness ratio varies from 0.49 to 1.24.

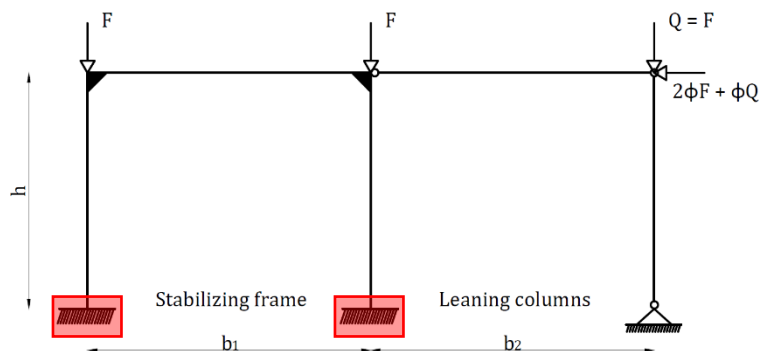


Figure 74: Clamped moment-resisting frame with a hinge-connected leaning column

Table 21: Input values for frame B to carry out GMNIA III and GMNIA IV

H-section	h	b ₁ =b ₂	F	Q	H	φ	λ	C	α _{cr}	L _{cr,LBA}	L _{cr,nomogram}	L _{cr,Yura}	Δ	A ₁	A ₂	A ₃	
[-]	[mm]	[mm]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[mm]	[mm]	[mm]	[mm]	[-]	[-]	[-]	
frame B01	HEA300	5000	5000	1000	1000	0	3.65E-03	0.73	1	7.05	7131	5783	7082	5.98	-6.12	-1.71	0
frame B02	HEA300	5000	7500	1000	1000	0	3.65E-03	0.77	1.5	6.38	7491	6111	7484	6.65	-6.12	-1.71	0
frame B03	HEA300	5000	10000	1000	1000	0	3.65E-03	0.80	2	5.87	7815	6397	7834	7.20	-6.12	-1.71	0
frame B1	HEA120	3000	3000	1000	1000	0	4.08E-03	1.12	1	0.68	4200	3470	4249	3.43	-3.44	-0.89	-1.19
frame B2	HEA120	3000	4500	1000	1000	0	4.08E-03	1.18	1.5	0.61	4429	3667	4491	3.87	-3.44	-0.89	-1.19
frame B3	HEA120	3000	6000	1000	1000	0	4.08E-03	1.24	2	0.56	4629	3838	4701	4.22	-3.44	-0.89	-1.19
frame B4	HEA160	3000	3000	1000	1000	0	4.08E-03	0.84	1	1.84	4238	3470	4249	3.51	-3.44	-0.89	-1.19
frame B5	HEA160	3000	4500	1000	1000	0	4.08E-03	0.89	1.5	1.66	4461	3667	4491	3.93	-3.44	-0.89	-1.19
frame B6	HEA160	3000	6000	1000	1000	0	4.08E-03	0.93	2	1.52	4658	3838	4701	4.27	-3.44	-0.89	-1.19
frame B7	HEA220	3000	3000	1000	1000	0	4.08E-03	0.62	1	5.71	4339	3470	4249	3.70	-3.68	0	0
frame B8	HEA220	3000	4500	1000	1000	0	4.08E-03	0.65	1.5	5.20	4545	3667	4491	4.08	-3.68	0	0
frame B9	HEA220	3000	6000	1000	1000	0	4.08E-03	0.68	2	4.79	4735	3838	4701	4.40	-3.68	0	0
frame B10	HEA280	3000	3000	1000	1000	0	4.08E-03	0.49	1	13.44	4478	3470	4249	3.96	-3.68	0	0
frame B11	HEA280	3000	4500	1000	1000	0	4.08E-03	0.51	1.5	12.39	4663	3667	4491	4.28	-3.68	0	0
frame B12	HEA280	3000	6000	1000	1000	0	4.08E-03	0.53	2	11.49	4842	3838	4701	4.56	-3.68	0	0

A different mechanical scheme means also a different distribution of the internal forces. In the case of frame A, the acting critical bending moment is the height times the horizontal load divided over the two columns, as shown on the left in Figure 75. When the bottom ends of the column are clamped, as is the case for frame B, then the critical bending moment reduces due to the bending moment resistance of the bottom end. However, the value for the bending moment is harder to determine since frame B is a statically indeterminate frame with the 3rd degree. Implementing the statically indeterminate frame in the calculation, and still be able to solve for the ultimate load takes a lot of effort. For that reason, linear interpolation over the results of computer software MatrixFrame is used to calculate a boundary factor (*bf* in Figure 75) to reduce the bending moment. Value *bf* is obtained by applying a horizontal load of 100 kN in the computer software and then dividing the $M_{Ed,software}$ of frame B by the $M_{Ed,software}$ of frame A.

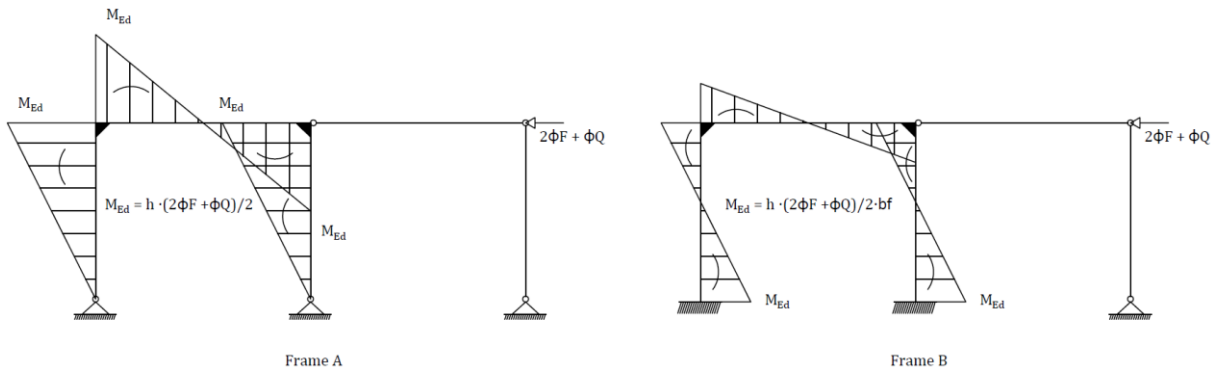


Figure 75: Bending moment diagram for the pin-ended frame (left) and clamped frame (right)

7.4 RESULTS

7.4.1 Frame A, A-NL, A-V, and B

The results are bundled for the frame configurations without and with the additional horizontal load. In this section, the results of the frame configurations without the additional horizontal load are given, which are frames A, A-NL, A-V, and B. The results of the frames are processed in the same way as explained in chapter 6, meaning that for each frame $F_{ult,i}$ and $N_{ult,i}$ are performed as in section 6.4 and 6.5 respectively, and converted into a unity check as in section 6.8. The results for frame A, A-NL, A-V, and B are given in Table 24. For all corresponding values see appendix C.

Section 6.8 concluded that the Yura approach gives the most accurate result for frame 01 compared to the ultimate resistance load with the most realistic input (LBA). The values in Table 24 indicates that this is also the case for all other frames. For that reason, all the results with the Yura approach are compared to the most realistic approach in Table 22.

Table 22: Difference between $U.C._{EC,Yura}$ and $U.C._{EC,LBA}$ $\left(\frac{U.C._{EC,Yura}-U.C._{EC,LBA}}{U.C._{EC,LBA}}\right)$ for frames A, A-NL, A-V, and B

frame 01	-1.06%	frame 01-NL	1.01%	frame 01-V	-4.85%	frame B01	0.36%
frame 02	-1.35%	frame 02-NL	0.37%	frame 02-V	-4.27%	frame B02	0.05%
frame 03	-1.35%	frame 03-NL	0.05%	frame 03-V	-3.58%	frame B03	-0.16%
frame A1	-2.39%	frame A1-NL	0.29%	frame A1-V	-6.47%	frame B1	-1.26%
frame A2	-2.15%	frame A2-NL	-0.08%	frame A2-V	-5.22%	frame B2	-1.58%
frame A3	-1.87%	frame A3-NL	-0.24%	frame A3-V	-4.24%	frame B3	-1.83%
frame A4	-1.69%	frame A4-NL	0.67%	frame A4-V	-5.64%	frame B4	-0.18%
frame A5	-1.74%	frame A5-NL	0.14%	frame A5-V	-4.74%	frame B5	-0.51%
frame A6	-1.61%	frame A6-NL	-0.11%	frame A6-V	-3.91%	frame B6	-0.74%
frame A7	-0.10%	frame A7-NL	1.43%	frame A7-V	-3.48%	frame B7	0.79%
frame A8	-0.64%	frame A8-NL	0.73%	frame A8-V	-3.33%	frame B8	0.50%
frame A9	-0.79%	frame A9-NL	0.39%	frame A9-V	-2.86%	frame B9	0.33%
frame A10	1.20%	frame A10-NL	1.79%	frame A10-V	-0.83%	frame B10	1.27%
frame A11	0.40%	frame A11-NL	1.10%	frame A11-V	-1.50%	frame B11	0.99%
frame A12	0.08%	frame A12-NL	0.75%	frame A12-V	-1.45%	frame B12	0.84%

7.4.2 Frame A-H, A-NLH, and A-VH

In this section, the results of the frame configurations with the additional horizontal load are given, which are frames AH, A-NLH, and A-VH. The results for frame AH, A-NLH, A-VH are given in Table 25. For all corresponding values see appendix C. The difference between the Yura approach and the most realistic approach (LBA) are given in Table 23.

Table 23: Difference between $U.C._{EC,Yura}$ and $U.C._{EC,LBA}$ $\left(\frac{U.C._{EC,Yura}-U.C._{EC,LBA}}{U.C._{EC,LBA}}\right)$ for frames A-H, A-NLH and A-VH

frame 01-H	-1.05%	frame 01-VH	-4.83%	frame 01-NLH	1.00%
frame 02-H	-1.33%	frame 02-VH	-4.24%	frame 02-NLH	0.36%
frame 03-H	-1.31%	frame 03-VH	-3.55%	frame 03-NLH	0.05%
frame A1-H	-2.35%	frame A1-VH	-6.44%	frame A1-NLH	0.28%
frame A2-H	-2.09%	frame A2-VH	-5.19%	frame A2-NLH	-0.08%
frame A3-H	-1.81%	frame A3-VH	-4.21%	frame A3-NLH	-0.23%
frame A4-H	-1.66%	frame A4-VH	-5.61%	frame A4-NLH	0.66%
frame A5-H	-1.71%	frame A5-VH	-4.71%	frame A5-NLH	0.14%
frame A6-H	-1.58%	frame A6-VH	-3.88%	frame A6-NLH	-0.10%
frame A7-H	-0.10%	frame A7-VH	-3.47%	frame A7-NLH	1.43%
frame A8-H	-0.63%	frame A8-VH	-3.31%	frame A8-NLH	0.73%
frame A9-H	-0.78%	frame A9-VH	-2.84%	frame A9-NLH	0.38%
frame A10-H	1.20%	frame A10-VH	-0.83%	frame A10-NLH	1.80%
frame A11-H	0.40%	frame A11-VH	-1.49%	frame A11-NLH	1.11%
frame A12-H	0.08%	frame A12-VH	-1.44%	frame A12-NLH	0.75%

Table 24: Summary of the outputs and results for frames A, A-NL, A-V, and B

	λ	C	$F_{ult,GMNIA}$	$N_{ult,NEN\&}$	$N_{ult,EC}$	$N_{ult,EC}$	$N_{ult,EC}$	U.C.	U.C.	U.C.	U.C.	$F_{ult,GMNIA}$
			IV	modEC,	nomogram	LBA	Yura	NEN&modEC,	EC,nomogram	EC,LBA	EC,Yura	III
			nomogram				nomogram					
	[-]	[-]	[kN]	[kN]	[kN]	[kN]	[kN]	[-]	[-]	[-]	[-]	[kN]
frame 01	1.45	1	1671.9	1447.6	1580.6	1230.6	1217.6	0.87	0.95	0.74	0.73	1604.6
frame 02	1.55	1.5	1503.3	1339.9	1462.1	1123.6	1108.4	0.89	0.97	0.75	0.74	1449.3
frame 03	1.64	2	1355.0	1244.4	1356.2	1028.4	1014.5	0.92	1.00	0.76	0.75	1311.5
frame A1	2.25	1	165.5	172.0	184.3	135.6	132.4	1.04	1.11	0.82	0.80	160.5
frame A2	2.41	1.5	145.8	155.6	166.3	121.0	118.4	1.07	1.14	0.83	0.81	142.0
frame A3	2.56	2	129.6	141.7	151.1	109.0	106.9	1.09	1.17	0.84	0.83	126.8
frame A4	1.68	1	441.5	409.5	444.9	338.4	332.7	0.93	1.01	0.77	0.75	425.6
frame A5	1.80	1.5	392.7	375.5	407.3	306.0	300.6	0.96	1.04	0.78	0.77	380.0
frame A6	1.91	2	351.3	346.0	374.5	278.1	273.6	0.99	1.07	0.79	0.78	342.2
frame A7	1.22	1	1334.2	1032.5	1127.6	910.5	909.6	0.77	0.85	0.68	0.68	1227.5
frame A8	1.30	1.5	1215.9	968.5	1059.5	843.9	838.5	0.80	0.87	0.69	0.69	1133.7
frame A9	1.37	2	1103.0	909.9	996.1	781.6	775.5	0.82	0.90	0.71	0.70	1041.3
frame A10	0.96	1	2572.3	1937.0	2091.4	1790.0	1811.4	0.75	0.81	0.70	0.70	2346.8
frame A11	1.01	1.5	2518.6	1852.1	2008.8	1702.0	1708.9	0.74	0.80	0.68	0.68	2278.0
frame A12	1.07	2	2434.8	1771.3	1928.2	1611.2	1612.5	0.73	0.79	0.66	0.66	2197.3
frame 01-NL	1.20	1	2318.1	1649.1	1649.1	1632.7	1649.1	0.71	0.71	0.70	0.71	2142.7
frame 02-NL	1.28	1.5	2129.1	1519.9	1519.9	1514.4	1519.9	0.71	0.71	0.71	0.71	1989.3
frame 03-NL	1.35	2	1946.8	1405.2	1405.2	1404.4	1405.2	0.72	0.72	0.72	0.72	1835.9
frame A1-NL	1.87	1	235.7	190.1	190.1	189.5	190.1	0.81	0.81	0.80	0.81	228.9
frame A2-NL	1.99	1.5	209.8	170.8	170.8	171.0	170.8	0.81	0.81	0.81	0.81	204.5
frame A3-NL	2.11	2	188.1	154.8	154.8	155.2	154.8	0.82	0.82	0.82	0.82	184.4
frame A4-NL	1.40	1	623.6	462.8	462.8	459.7	462.8	0.74	0.74	0.74	0.74	590.4
frame A5-NL	1.48	1.5	562.5	422.0	422.0	421.4	422.0	0.75	0.75	0.75	0.75	537.4
frame A6-NL	1.57	2	508.5	386.7	386.7	387.1	386.7	0.76	0.76	0.76	0.76	489.2
frame A7-NL	1.01	1	1682.9	1180.2	1180.2	1163.6	1180.2	0.70	0.70	0.69	0.70	1540.4
frame A8-NL	1.07	1.5	1606.6	1105.8	1105.8	1097.7	1105.8	0.69	0.69	0.68	0.69	1467.1
frame A9-NL	1.13	2	1519.8	1036.8	1036.8	1032.8	1036.8	0.68	0.68	0.68	0.68	1387.2
frame A10-NL	0.79	1	2686.7	2190.3	2190.3	2151.8	2190.3	0.82	0.82	0.80	0.82	2538.6
frame A11-NL	0.84	1.5	2632.3	2101.5	2101.5	2078.6	2101.5	0.80	0.80	0.79	0.80	2543.0
frame A12-NL	0.88	2	2659.4	2014.6	2014.6	1999.6	2014.6	0.76	0.76	0.75	0.76	2477.1
frame 01-V	2.84	1	462.5	746.8	1179.8	375.7	357.4	1.61	2.55	0.81	0.77	453.9
frame 02-V	3.04	1.5	404.4	699.3	1112.9	334.1	319.9	1.73	2.75	0.83	0.79	398.4
frame 03-V	3.24	2	357.4	657.5	1051.3	299.6	288.8	1.84	2.94	0.84	0.81	353.5
frame A1-V	4.40	1	44.0	96.4	146.3	38.3	35.9	2.19	3.32	0.87	0.81	43.4
frame A2-V	4.73	1.5	38.2	89.2	134.9	33.6	31.9	2.34	3.54	0.88	0.84	37.7
frame A3-V	5.05	2	33.6	83.0	124.9	29.9	28.6	2.47	3.72	0.89	0.85	33.3
frame A4-V	3.29	1	119.7	214.9	336.7	99.8	94.2	1.80	2.81	0.83	0.79	117.7
frame A5-V	3.53	1.5	104.2	200.5	315.1	88.3	84.1	1.92	3.02	0.85	0.81	102.8
frame A6-V	3.77	2	92.0	187.8	295.5	78.9	75.8	2.04	3.21	0.86	0.82	91.1
frame A7-V	2.39	1	377.3	529.9	829.3	294.5	284.3	1.40	2.20	0.78	0.75	369.7
frame A8-V	2.55	1.5	331.9	498.8	790.8	264.3	255.5	1.50	2.38	0.80	0.77	326.8
frame A9-V	2.71	2	294.3	471.3	754.5	238.4	231.6	1.60	2.56	0.81	0.79	291.3
frame A10-V	1.88	1	908.9	1022.4	1528.2	652.6	647.2	1.12	1.68	0.72	0.71	885.5
frame A11-V	1.99	1.5	810.3	970.0	1478.7	596.5	587.6	1.20	1.82	0.74	0.73	795.3
frame A12-V	2.12	2	723.3	923.2	1430.8	544.8	536.9	1.28	1.98	0.75	0.74	714.6
frame B01	0.73	1	2781.6	2633.3	2792.7	2548.7	2558.0	0.95	1.00	0.92	0.92	3031.8
frame B02	0.77	1.5	2780.1	2555.0	2720.6	2466.1	2467.3	0.92	0.98	0.89	0.89	2778.2
frame B03	0.80	2	2909.0	2487.2	2657.1	2391.9	2388.1	0.86	0.91	0.82	0.82	2790.0
frame B1	1.12	1	579.9	428.0	465.5	389.2	384.3	0.74	0.80	0.67	0.66	607.1
frame B2	1.18	1.5	555.2	405.3	441.4	364.8	359.0	0.73	0.80	0.66	0.65	567.7
frame B3	1.24	2	533.7	386.3	421.1	344.6	338.3	0.72	0.79	0.65	0.63	531.9
frame B4	0.84	1	1156.4	831.4	890.8	793.8	792.3	0.72	0.77	0.69	0.69	1173.7
frame B5	0.89	1.5	1142.3	800.8	861.2	760.2	756.3	0.70	0.75	0.67	0.66	1160.2
frame B6	0.93	2	1126.6	774.6	835.4	730.9	725.5	0.69	0.74	0.65	0.64	1139.5
frame B7	0.62	1	2014.6	1660.2	1737.8	1620.6	1633.3	0.82	0.86	0.80	0.81	2016.4
frame B8	0.65	1.5	2021.2	1621.8	1704.3	1583.0	1591.0	0.80	0.84	0.78	0.79	2014.8
frame B9	0.68	2	2022.3	1588.3	1674.6	1548.4	1553.5	0.79	0.83	0.77	0.77	2008.7
frame B10	0.49	1	2828.4	2710.2	2790.1	2648.3	2681.9	0.96	0.99	0.94	0.95	2824.4
frame B11	0.51	1.5	2828.4	2664.9	2752.5	2609.9	2635.7	0.94	0.97	0.92	0.93	2818.7
frame B12	0.53	2	2828.4	2625.3	2719.2	2573.1	2594.7	0.93	0.96	0.91	0.92	2815.8

Table 25: Summary of the outputs and results for frames A, A-NL, A-V, and B

	λ	C	$F_{ult,GMNIA}$	$N_{ult,NEN\&}$	$N_{ult,EC}$	$N_{ult,EC}$	$N_{ult,EC}$	U.C.	U.C.	U.C.	U.C.	$F_{ult,GMNIA}$
			IV	modEC,	nomogram	LBA	Yura	NEN&modEC,	EC,nomogram	EC,LBA	EC,Yura	III
			nomogram		nomogram							
			[-]	[-]	[kN]	[kN]	[kN]	[kN]	[kN]	[-]	[-]	[-]
frame 01-H	1.45	1	1457.5	1328.4	1444.4	1127.7	1115.9	0.91	0.99	0.77	0.77	1426.7
frame 02-H	1.55	1.5	1241.7	1177.0	1276.9	985.9	972.9	0.95	1.03	0.79	0.78	1222.5
frame 03-H	1.64	2	1062.8	1045.8	1131.7	864.1	852.7	0.98	1.06	0.81	0.80	1045.7
frame A1-H	2.25	1	127.6	142.7	152.3	112.8	110.2	1.12	1.19	0.88	0.86	126.2
frame A2-H	2.41	1.5	100.8	116.7	124.0	91.1	89.2	1.16	1.23	0.90	0.89	99.1
frame A3-H	2.56	2	78.8	95.3	101.0	73.7	72.3	1.21	1.28	0.93	0.92	77.1
frame A4-H	1.68	1	381.3	373.1	403.9	308.4	303.2	0.98	1.06	0.81	0.80	373.1
frame A5-H	1.80	1.5	318.8	326.4	352.2	266.1	261.6	1.02	1.10	0.83	0.82	314.0
frame A6-H	1.91	2	268.1	286.7	308.4	230.7	227.1	1.07	1.15	0.86	0.85	265.7
frame A7-H	1.22	1	1227.7	991.2	1080.1	872.4	871.5	0.81	0.88	0.71	0.71	1145.8
frame A8-H	1.30	1.5	1084.7	910.7	992.9	791.8	786.8	0.84	0.92	0.73	0.73	1024.9
frame A9-H	1.37	2	958.1	838.1	913.3	718.4	712.8	0.87	0.95	0.75	0.74	912.7
frame A10-H	0.96	1	2564.7	1893.9	2042.7	1746.6	1767.6	0.74	0.80	0.68	0.69	2346.3
frame A11-H	1.01	1.5	2414.8	1789.7	1937.9	1640.3	1646.9	0.74	0.80	0.68	0.68	2215.3
frame A12-H	1.07	2	2246.0	1691.4	1836.8	1533.6	1534.7	0.75	0.82	0.68	0.68	2071.0
frame 01-VH	2.84	1	411.3	694.4	1089.8	348.7	331.9	1.69	2.65	0.85	0.81	406.4
frame 02-VH	3.04	1.5	342.3	625.9	986.1	298.5	285.8	1.83	2.88	0.87	0.84	335.8
frame 03-VH	3.24	2	287.3	565.8	892.4	257.3	248.2	1.97	3.11	0.90	0.86	282.7
frame A1-VH	4.40	1	34.0	81.3	122.3	32.4	30.3	2.39	3.59	0.95	0.89	33.7
frame A2-VH	4.73	1.5	26.4	68.4	102.0	25.8	24.5	2.59	3.87	0.98	0.93	25.8
frame A3-VH	5.05	2	20.3	57.3	84.7	20.7	19.8	2.82	4.18	1.02	0.98	19.7
frame A4-VH	3.29	1	104.2	198.3	308.7	92.1	86.9	1.90	2.96	0.88	0.83	103.4
frame A5-VH	3.53	1.5	85.7	177.3	276.0	78.0	74.3	2.07	3.22	0.91	0.87	84.7
frame A6-VH	3.77	2	71.0	159.0	246.9	66.7	64.1	2.24	3.48	0.94	0.90	70.1
frame A7-VH	2.39	1	353.0	512.2	799.0	284.2	274.3	1.45	2.26	0.81	0.78	346.6
frame A8-VH	2.55	1.5	302.1	473.8	747.4	250.6	242.3	1.57	2.47	0.83	0.80	296.8
frame A9-VH	2.71	2	260.3	439.9	699.3	222.1	215.8	1.69	2.69	0.85	0.83	255.3
frame A10-VH	1.88	1	871.5	1003.2	1497.1	639.1	633.9	1.15	1.72	0.73	0.73	851.6
frame A11-VH	1.99	1.5	764.2	942.6	1433.2	578.3	569.7	1.23	1.88	0.76	0.75	751.9
frame A12-VH	2.12	2	671.9	888.3	1371.8	522.8	515.3	1.32	2.04	0.78	0.77	664.1
frame 01-NLH	1.20	1	2020.4	1503.3	1503.3	1488.4	1503.3	0.74	0.74	0.74	0.74	1907.4
frame 02-NLH	1.28	1.5	1750.3	1323.3	1323.3	1318.5	1323.3	0.76	0.76	0.75	0.76	1668.3
frame 03-NLH	1.35	2	1519.2	1168.5	1168.5	1167.9	1168.5	0.77	0.77	0.77	0.77	1452.3
frame A1-NLH	1.87	1	185.3	156.8	156.8	156.3	156.8	0.85	0.85	0.84	0.85	181.1
frame A2-NLH	1.99	1.5	146.0	127.2	127.2	127.3	127.2	0.87	0.87	0.87	0.87	142.1
frame A3-NLH	2.11	2	114.6	103.2	103.2	103.5	103.2	0.90	0.90	0.90	0.90	112.2
frame A4-NLH	1.40	1	535.9	419.3	419.3	416.5	419.3	0.78	0.78	0.78	0.78	517.1
frame A5-NLH	1.48	1.5	447.6	364.0	364.0	363.5	364.0	0.81	0.81	0.81	0.81	437.0
frame A6-NLH	1.57	2	388.4	317.6	317.6	317.9	317.6	0.82	0.82	0.82	0.82	377.5
frame A7-NLH	1.01	1	1556.3	1128.8	1128.8	1112.9	1128.8	0.73	0.73	0.72	0.73	1443.0
frame A8-NLH	1.07	1.5	1423.3	1034.2	1034.2	1026.7	1034.2	0.73	0.73	0.72	0.73	1327.7
frame A9-NLH	1.13	2	1287.2	948.3	948.3	944.7	948.3	0.74	0.74	0.73	0.74	1214.0
frame A10-NLH	0.79	1	2806.6	2137.8	2137.8	2099.9	2137.8	0.76	0.76	0.75	0.76	2652.5
frame A11-NLH	0.84	1.5	2739.7	2025.0	2025.0	2002.7	2025.0	0.74	0.74	0.73	0.74	2563.9
frame A12-NLH	0.88	2	2636.8	1916.1	1916.1	1901.7	1916.1	0.73	0.73	0.72	0.73	2465.9

7.5 CONCLUSIONS

7.5.1 Frame A, A-NL, A-V, and B

The results from Table 24 can be plotted into a unity check graph, where each symbol indicates a different frame configuration. First, the unity check graphs are given for the NEN 6771 & modified EN 1993-1-1 and the current EN 1993-1-1 in Figure 76, left and right respectively. It can be seen that the EN 1993-1-1 many frame configurations lie in the unsafe region. This was expected since the current EN 1993-1-1 does not take into account the leaning column effect at all. The unsafe results start at about a slenderness ratio of 1.5 and go up to a unity check of 3.7 for a slenderness ratio of 5.0. As discussed before, the NEN 6771 considered the leaning column effect in the load

side of the equation (F_{tot}). It can be stated from Figure 76 that the NEN 6771 & modified EN 1993-1-1 gives lower ultimate design resistance loads than the current EN 1993-1-1, which is (almost) safe enough for frame A and B, but in the case of frame A-V with the increased leaning column effect it gives unsafe results. Unsafe results start at about a slenderness ratio of 2 and increase somewhat linearly up to a unity check of 2.5 for a slenderness ratio of 5.0.

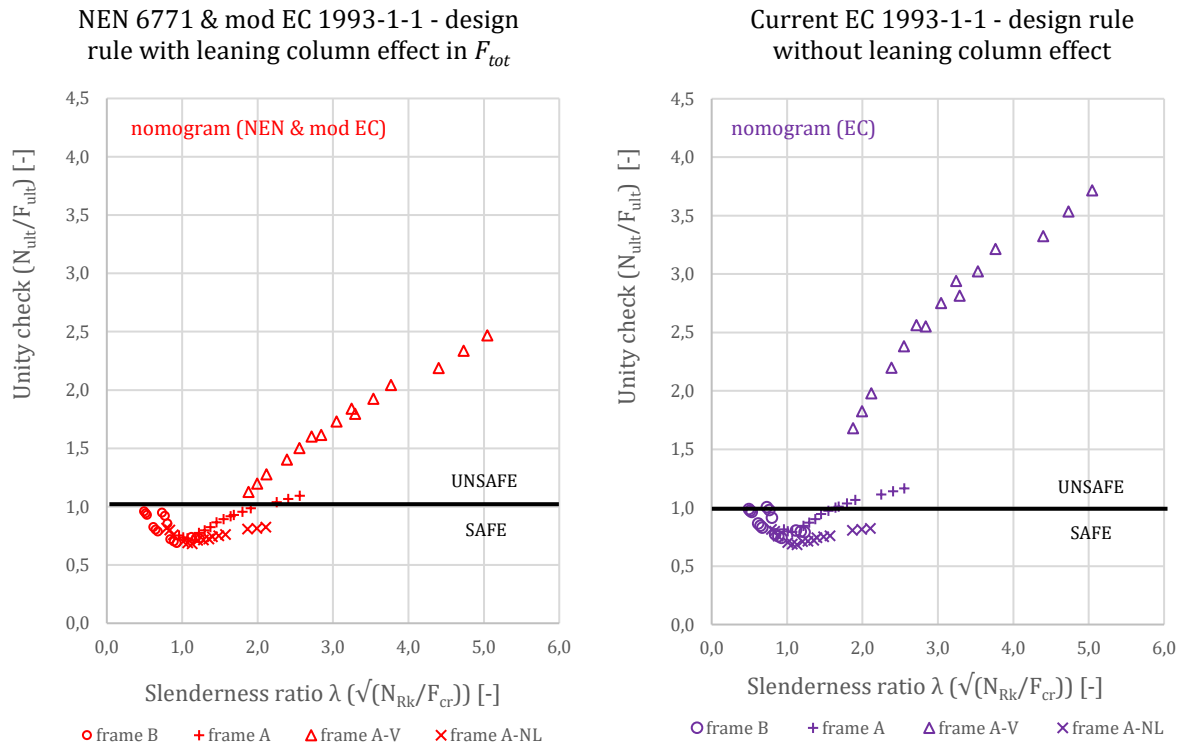


Figure 76: Unity check graphs for the NEN 6771& mod EN 1993-1-1 (left) and current EN 1993-1-1 (right) for frames A, A-NL, A-V, and B

Next, the unity check graph is given for the EN 1993-1-1 design rule where the leaning column effect is included in L_{cr} for frames A, A-NL, A-V, and B. In Figure 77 the results are shown for the EN 1993-1-1 design rule in combination with L_{cr} obtained from the LBA (most realistic) and with the Yura approach. The sentence new L_{cr} is used in the title of Figure 77 since the current nomogram does not include the leaning column effect in L_{cr} . Figure 77 shows that including the leaning column effect in L_{cr} leads to safe results for all cases. When we compare the results to Figure 76, it can be seen that all the results of the EN 1993-1-1 with the new L_{cr} are lower than the results from the NEN 6771 & modified EN 1993-1-1 and the current EN 1993-1-1 (specific values in Table 24). This means that results for the frames that are safe in the case of the NEN 6771 & modified EN 1993-1-1 and the current EN 1993-1-1 will be more conservative when considering the EN 1993-1-1 with the new L_{cr} . This increase in conservatism is different per frame configuration.

Section 6.7.1.4 showed that $N_{ult,EC,Yura}$ for frame 01 only differ -1.06% to $N_{ult,EC,LBA}$, where $N_{ult,EC,LBA}$ is performed with the most realistic input obtained from the FE-model and assumed as the most realistic design-resistance load. This means that for frame 01 the Yura approach is very accurate. When this difference is calculated for all frames, as given in Table 22, it can be seen that this is the case for all frames. The lowest difference has a value of -6.47% and the highest difference a value of 1.79%. The mean of all differences is -1.07%. The mean indicates that on average the results with Yura are lower than the most realistic one, which leads to safer results. Considering the above

gives that using the Yura approach in combination with the EN 1993-1-1 design rule give for all cases accurate and safe results.

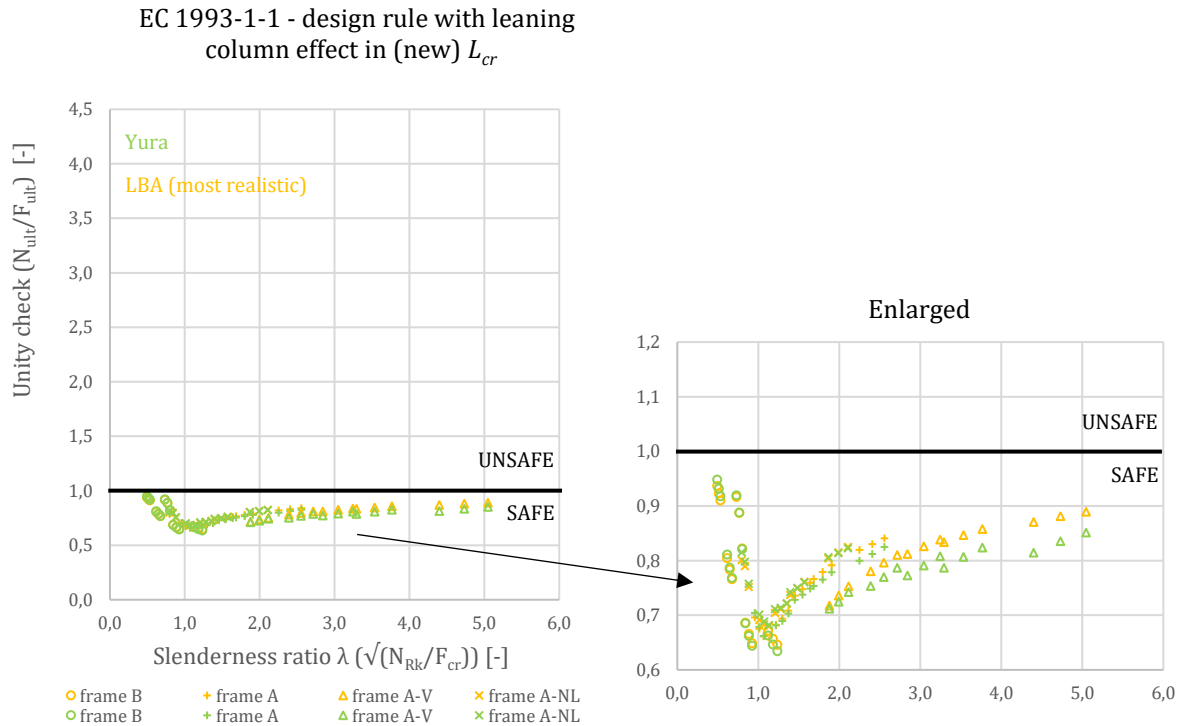


Figure 77: Unity check graph for the EN 1993-1-1 in combination with the LBA and Yura for frames A, A-NL, A-V, and B

As discussed before Frame A-NL is included to check whether the ratios between N_{ult} and F_{ult} are reliable. Due to the absence of the leaning column effect in frame A-NL, the slenderness ratio $\bar{\lambda}$ are lower than compared to frame A, as can be seen in Table 24. However, the ratios between N_{ult} and F_{ult} for frame A-NL are comparable to the ratios for frames A and B, which have a comparable slenderness ratio. This is the case for both Figure 76 and Figure 77, meaning that this validates the other results.

7.5.2 Frames A-H, A-NLH, and A-VH

As already explained in section 7.3.4, frames A-H, A-NLH, and A-VH are only extended with an additional horizontal load and this means that for the ultimate resistance loads only M_{Ed} increases in equation (2) and (6), while in the FE-model the actual additional horizontal displacement is also taken into account. When we look at the unity checks as given in Table 24 and Table 25, it can be seen that the additional horizontal load influences the results, since all the unity checks found in Table 25 are higher than in Table 24. For example, when we consider the unity checks in Table 25 for frame 01-H, they are all higher than the unity checks for frame 01 in Table 24. This means that the additional horizontal displacement is indeed not taken into account in the design rules and therefore decreases the conservatism margins.

Since section 7.5.1 already showed that the NEN 6771 & modified EN 1993-1-1 and the current EN 1993-1-1 lead to unsafe results, the results for these design rules will not be plotted in a unity check graph, since it will only show that the results are even more unsafe. In the case of the EN 1993-1-1 design rule where the leaning column effect is included in L_{cr} , the decrease of the conservatism margin can result in unsafe unity checks, due to the already small margin for the slender frame configurations. When we plot the unity check graph for the EN 1993-1-1 design rule where the leaning column effect is included in L_{cr} for frames A-H, A-NLH, and A-VH (Figure 78), we can see that the conservatism margin decreased and for the most slender frame even led to an

unsafe result when considering the LBA. However, the small difference between the results with LBA and Yura as found in Table 23 (-4.21%), makes that Yura does lead to a safe result. The lowest difference in Table 23 has a value of -6.44% and the highest difference a value of 1.80%. The mean of all differences is -1.39%, which again leads to safe results.

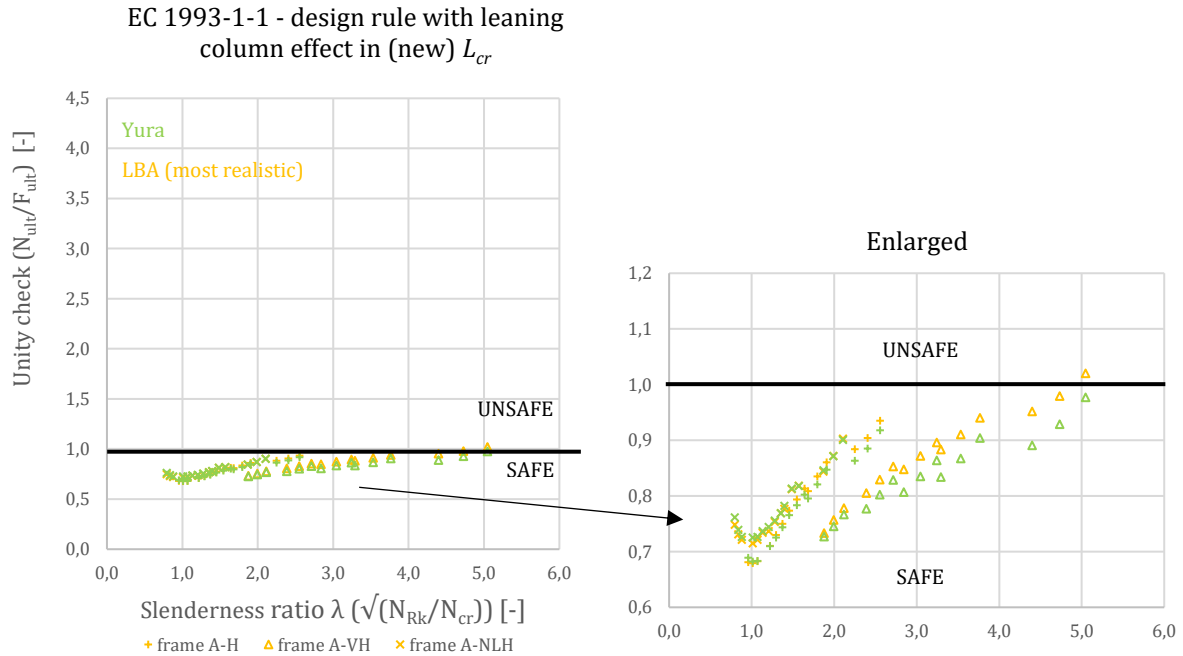


Figure 78: Unity check graph for the EN 1993-1-1 in combination with the LBA and Yura for frames A-H, A-NLH, and A-H

7.5.3 Differences between GMNIA III and GMNIA IV

As already discussed in section 6.6.1 is only GMNIA IV considered in chapter 7. The imperfections of GMNIA IV are performed according to Shayan et al. [9]. In their paper, they state that the use of only the first critical eigenmode (which is the case for GMNIA III) leads to conservative results. However, to validate GMNIA IV the differences between GMNIA IV and GMNIA III are plotted in Figure 79. The differences between GMNIA III and GMNIA IV are close to each other with a maximum difference of 10%. The difference is the largest in the region where the most conservative results lay, namely between $\bar{\lambda}=0.5-2$, meaning that this difference does not influence the end conclusion.

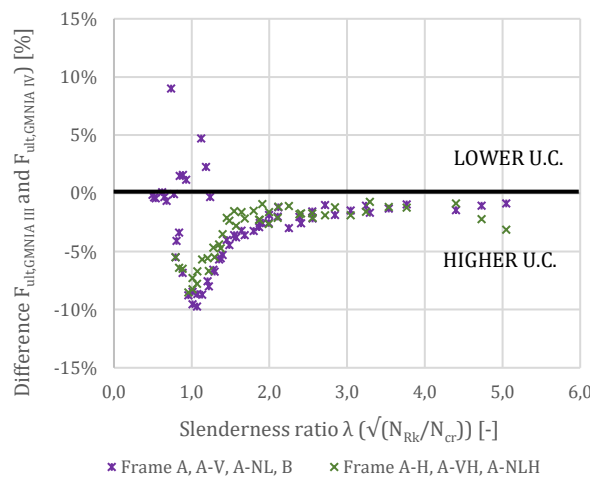


Figure 79: Differences between GMNIA III and GMNIA IV, plotted along with the slenderness ratio

8 CONCLUSIONS AND RECOMMENDATIONS

8.1 CONCLUSIONS

In The Netherlands, in the past, the NEN 6771 provided the design rules about the stability of steel columns loaded in compression and bending. It was found that the design rules of the NEN 6771 include the leaning column effect in the load side of the equation with the value F_{tot} . In 2012, the Eurocode was adopted in The Netherlands as a replacement for the NEN series. The Eurocode (EC) is a European standard to assess the structural safety of all possible building structures. The EC EN 1993-1-1 is the basic part containing the design rules for steel structures. It was found that the design rules of the EN 1993-1-1 do not include the leaning column effect at all. For that reason, the goal of this research is to numerically verify the design rule of the NEN 6771 for the leaning column effect, since the background of the NEN 6771 about the leaning column effect is unknown, and validate and rewrite the design rule of the EN 1993-1-1 for a more clear presentation and content regarding the leaning column effect.

In the process of this research, some decisions were made that did not reflect reality precisely. These decisions were about modeling the cross-section without the root radius and the high residual stresses for steel quality's higher than S235. In the process, it turned out that these decisions influence the results barely.

The research scope aimed to cover all the cases where the leaning column effect is governing and reflects a large range of leaning column problems. The parameter that is used to reflect the range is the slenderness ratio, defined by $\bar{\lambda} = \sqrt{N_{Rk}/F_{cr}}$. The frame configurations were studied extensively by carrying out various geometric and material non-linear analyses with imperfections (GMNIA) with a parametric Finite Element (FE) model and comparing them to various ultimate resistance outcomes from the design rules. This extensive study showed that some margins between GMNIA IV, the simulation closest to reality, and the design rule of the EN 1993-1-1 are unsafe. For a slenderness ratio greater than 1.5, the current EN 1993-1-1 design rule gives unsafe results. The NEN 6771 design rule, also known as the design rule where the leaning column effect is included in the load side of the equation, decreased the ultimate resistance load, but not enough. At a slenderness ratio greater than 2, the design rule of the NEN 6771 is also not safe anymore. This can be explained by the fact that the design rule considers the member problem more as a function of an individual column (P- δ effect), instead of a function of all columns in the story (P- Δ effect). The lateral deflection (Δ) that increases the bending moment is predicted inaccurately, which leads to unsafer results when the lateral deflection increases. This can be seen by the linear increase of the unity checks along with the slenderness ratio. Different behavior is spotted for frame B, the frame with the clamped column ends, which can not be explained.

Authorities asked whether it is possible to rewrite the current EN 1993-1-1 design rule since it is hard to check for what value for the leaning column effect an engineer has accounted for, or to see it has been taken into account at all. It is possible to rewrite the design rule of the NEN 6771 into EN 1993-1-1 format. When the derivation of the EN 1993-1-1 design rule is carried again from the starting point, but now including a new value F_{lean} , the following equation is obtained:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \left(\bar{\lambda}^2 - \frac{1}{\chi} \right) (\chi - 1) \frac{F_{lean}}{N_{Rd}} + k \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \text{ where } k = \frac{1}{1 - \frac{\chi \cdot N_{Ed}}{F_{cr}}}$$

F_{lean} represents the additional load on the frame due to the leaning column effect, instead of the total load on the structure as F_{tot} does in the NEN 6771 design rule. The modified EN 1993-1-1 equation gives the same results as the design rule of the NEN 6771, as intended. However, the equation has no use since the results of the NEN 6771 are unsafe.

Due to the unsafe result of the NEN 6771, different ways to include the leaning column effect in the design rules are studied. Geschwindner [1] wrote a paper about several ways to include the leaning column effect in the buckling length. These approaches are studied, and the literature study showed that several of these approaches overlap. For that reason, only two approaches are extensively investigated, namely the approach of Geschwindner and Yura. The results of the approaches can be related to the results of the FE-model since the α_{cr} obtained from the LBA can be used to determine an L_{cr} that reflects reality in the best way. In the case of Geschwindner, the American nomogram equation to determine the buckling length is extended to include the leaning column effect. However, it is found that this approach is not very accurate to the results obtained with the LBA. The Yura approach is based on the equilibrium of the structure. By equating the Euler buckling equation where the leaning column effect is not taken into account to the Euler buckling equation where the leaning column effect is included, the following equation is found:

$$\beta_{Yura} = \beta_{nomogram} \cdot \sqrt{\frac{\sum F + \sum Q}{\sum F}}$$

Where β_{Yura} is the modified effective length factor which takes into account the leaning column effect and $\beta_{nomogram}$ the modified effective length factor found by the nomogram in the Dutch National Annex NB.NA, which does not take into account the leaning column effect. The research showed that the use of β_{Yura} gives similar buckling lengths as the LBA, which is assumed as most realistic, and for that reason is used in the parametric study.

For all the slenderness ratios considered in the parametric study, the current EN 1993-1-1 design rule in combination with the Yura approach gives safe results. Besides that, also the differences between the results of the Yura approach and the LBA are low. The lowest difference between the ultimate resistance loads obtained with LBA and Yura has a value of -6.47% and the highest difference a value of 1.79%. The mean of all differences is -1.07%. The negative mean indicates that on average ultimate resistance loads with Yura are lower than the most realistic one (LBA), which leads to a higher, and therefore safer, conservatism margin.

For the design rules, a higher horizontal load only increases the bending moment M_{Ed} , while in the FE-model also the actual additional horizontal displacement is taken into account. For that reason, the increase of the horizontal load influences the conservatism margins, and in the case of the most slender structure with the LBA, the conservatism margin is reduced to even a negative value, meaning that the result for this frame is unsafe with a unity check of 1.02. However, the (small) difference between Yura and LBA gives that with considering the Yura approach, the frame is safe with a unity check of 0.98.

The research shows that a modified buckling length equation is more effective than a modified EN 1993-1-1 equation for including the leaning column effect. The modified buckling length equation is easy to implement since the current way of determining β with the Dutch National Annex can be kept, and only has to be extended with the square root of the sum of the load on the total structure (\sum stabilizing load $F + \sum$ leaning column load Q) divided by the sum of the load on the stabilizing structure (\sum stabilizing load F), or in other terms $\sqrt{(\sum F + \sum Q)/\sum F}$.

8.2 RECOMMENDATIONS

To apply the Yura approach in the Eurocode more research has to be done since only the most simplistic frame configuration has been studied. To ensure the correctness of the Yura approach, more complicated frame configurations should be studied and analyzed with FEM, for example, frames with more stories and more rows of attached leaning columns. In the case of frame A-V, the model should represent a stabilizing frame with a row of ten columns attached to it, without actual modeling these columns. It could be the case that when modeling the columns, the frame behaves differently. In the case of more stories, the stiffness of the joint is an important aspect, which can not be simulated by only adjusting the height of the simplistic frame configuration.

The additional horizontal wind load of 1 kN/m² is based on a broad assumption. It is recommended to apply a more accurate wind load for each frame height, since the slender frames are reasonable affected by the wind load. For more complicated frames this also can lead to high wind loads. When unsafe results are found related to higher wind loads in combination with the leaning column effect, it is advised to study the design rules regarding the lateral deflection of the structure. Currently, the lateral deflection is not predicted accurately. It is expected that the amplification factor $(\frac{\alpha_{cr}}{\alpha_{cr}-1})$ and the slope ($\phi = \phi_0 \cdot \alpha_h \cdot \alpha_m$) in the EC EN 1993-1-1 cause this.

In this research, the load is only applied to the columns. It is recommended to check the influence on the results when the beams are loaded, and failure of the beam is the first failure mode. In practice, the load is applied to the beam, instead of only on the column. This can influence the behavior of the structure.

9 BIBLIOGRAPHY

- [1] L. F. Geschwindner, "A Practical Look at Frame Analysis, Stability and Leaning Columns," *Engineering Journal*, vol. Fourth Quarter, pp. 167-181, 2002.
- [2] L. Simões da Silva, R. Simões and H. Gervásio, "Eurocode 3: Design of Steel Structures; Part 1-1: General rules and rules for buildings," ECCS Eurocode Design Manuals, Mem Martins, Portugal, 2010.
- [3] H. Snijder, F. Bijlaard and H. Steenbergen, "Valkuilen in de TGB Staal," *Bouwen met staal* 125, pp. 19-24, juli/augustus 1995.
- [4] N. Boissonnade, R. Greiner, J. Jaspart and J. Lindner, Book: Rules for Member Stability in EN 1993-1-1; Background documentation and design guidelines, Portugal: ECCS, 2006.
- [5] H. Snijder and H. Steenbergen, Stabiliteit 3; Staven belast op druk en buiging, unpublished.
- [6] K. Loorits, "Classification of cross sections for steel beams in different design codes," *Rakenteiden Mekaniikka, Vol 28*, vol. 28, pp. 19-33, 1995.
- [7] Comité Européen de Normalisation, "Eurocode 3: Design of steel structures- Part 1-1: General rules and rules for buildings," CEN, Brussels, 2005.
- [8] U. Vogel and e. al., "Ultimate Limit State Calculation of Sway Frames with Rigid Joints. Publication No 33," ECCS, Brussels, 1984.
- [9] S. Shayan, K. J. R. Rasmussen and H. Zhang, "On the modelling of initial geometric imperfections of steel frames in advanced analysis," *Journal of Constructional Steel Research* 98, pp. 167-177, 2014.
- [10] S. Taris, "MASTER THESIS - Stability of unbraced laterally supported steel sway-frames - applicability and scope of the general method of Eurocode 3," Technical University of Eindhoven, Eindhoven, 2009.
- [11] Z. Zhuang, Z. Liu, J. Liao and B. Cheng, "Extended Finite Element Method," Elsevier Inc., Beijing, 2014.
- [12] Abaqus-docs, "Choosing a shell element," Abaqus, 2012. [Online]. Available: <https://abaqus-docs.mit.edu/2017/English/SIMACAEELMRefMap/simaelm-c-sellelem.htm>. [Accessed 06 May 2020].
- [13] Abaqus-docs, "Configuring a static, general procedure," Abaqus, 2012. [Online]. Available: <https://abaqus-docs.mit.edu/2017/English/SIMACAECAERefMap/simacae-t-simconfigurestatic.htm>. [Accessed 19 May 2020].
- [14] Abaqus-docs, "Eigenvalue buckling prediction," Abaqus, 2012. [Online]. Available: <https://abaqus-docs.mit.edu/2017/English/SIMACAEANLRefMap/simaanl-c-eigenbuckling.htm>. [Accessed 07 May 2020].

- [15] Abaqus-docs, "Coupling constraints," Abaqus, 2012. [Online]. Available: <https://abaqus-docs.mit.edu/2017/English/SIMACAECSTRefMap/simacst-c-coupling.htm>. [Accessed 07 May 2020].
- [16] American Institute of Steel Construction, Inc. (AISC), "Load and Resistance Factor Design Specification for Structural Steel Buildings," AISC, Chicago, 1999.

APPENDIX A – LITERATURE AND INPUTS FOR THE DESIGN CODES

**APPENDIX A:
LITERATURE AND INPUTS FOR THE
DESIGN CODES**

TABLE OF CONTENTS

<i>1</i>	<i>VARIOUS NOMOGRAM SOLUTIONS.....</i>	<i>2</i>
1.1	Dutch Annex NB.NA.....	2
1.2	Modified nomograph equation.....	3
1.3	Yura Approach	4
1.4	Lim & McNamara approach.....	5
1.5	LeMessurier approach.....	5
1.6	American Institute of Steel Construction.....	6
<i>2</i>	<i>LITERATURE REGARDING EC 1993-1-1.....</i>	<i>6</i>
2.1	Members in compression	6
2.2	Members in bending and axial compression.....	8
2.2.1	Elastic flexural buckling.....	9
2.2.2	Elastic-plastic flexural buckling without lateral torsional buckling	11
<i>3</i>	<i>Bibliography.....</i>	<i>15</i>

1 VARIOUS NOMOGRAM SOLUTIONS

1.1 DUTCH ANNEX NB.NA

The NEN 6770 offers two nomographs where the solution for L_{cr} can be derived from. These nomographs are shown in Figure 1, where the left nomograph covers non-sway frames and the right nomograph sway frames. [1]

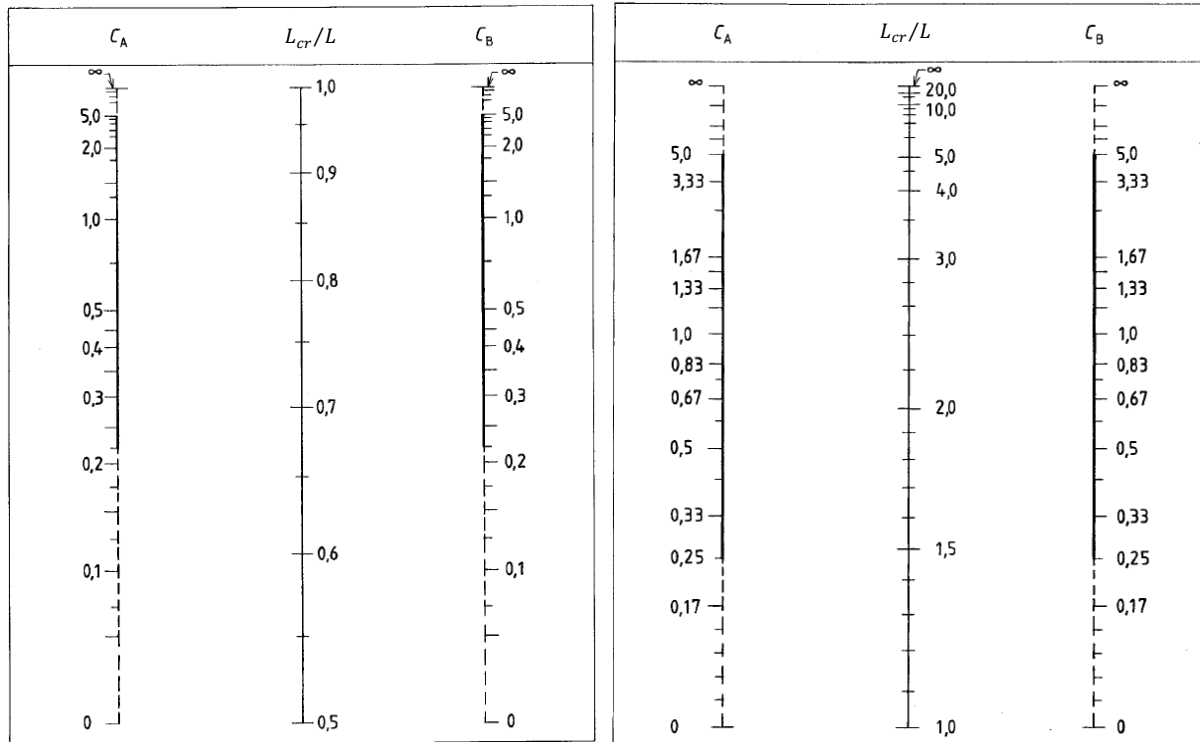


Figure 1: Nomogram for buckling length (L_{cr}) non-sway (left) and sway (right) frames

Value L_{cr} can be obtained from the nomograms by deriving flexibility parameters C_A and C_B , where the indices A and B refer to the two member-ends of the considered column. The flexibility parameters C_A and C_B can be derived with equation (28)

$$C_i = \frac{\sum \frac{I_{ctn}}{L_{ctn}}}{\sum \mu \frac{I_{bm}}{L_{bm}}} \quad (1)$$

Where I_{ctn} and I_{bm} are the second moments of inertia of the column and the connected beam respectively, L_{ctn} and L_{bm} are the length of the column and the connected beam respectively, and μ a correction factor. This correction factor μ takes the boundary condition at the other end of the supporting beam into account and is different between a non-sway and sway frames.

For non-sway frames:

μ is 2 if the other end is rigidly connected to one or more columns which are also loaded in compression;

μ is 3 if the other end is pinned;

μ is 4 if the other end is clamped.

For sway frames:

μ is 3 if the other end is pinned;

μ is 4 if the other end is clamped;

μ is 6 if the other end is rigidly connected to one or more columns which are also loaded in compression.

1.2 MODIFIED NOMOGRAPH EQUATION

Equation (2) gives the solution for the critical buckling load of the perfect column. In reality, perfectly pinned ends in a steel structure do not exist. Therefore equation (2) is extended as the following:

$$F_{cr} = \frac{(\pi^2 EI)}{(K_{exact} \cdot L_{cr})^2} \quad (2)$$

Where the K-factor is a mathematic adjustment on the capacity of the column, aiming for a better representation of reality. According to G.L. Geschwindner (2002) [2] the American codes for structural safety, LRFD, and AISC [3] specifications make use of a nomograph to determine the factor K. G.L. Geschwindner (2002) [2] states that the equation where this nomograph is based on is as follows:

$$\frac{G_A G_B (\pi/K)^2 - 36}{6(G_A + G_B)} = \frac{\pi/K}{\tan(\pi/K)} \quad (3)$$

Where A and B refer to the ends of the column and where:

$$G = \frac{\sum I_c / L_c}{\sum I_b / L_b} \quad (4)$$

To include the leaning column effect in equation (3), Figure 2 will be considered. For equilibrium of the leaning column CD, load Q must be balanced with a horizontal force with a magnitude of $Q\Delta/L$. This horizontal force must then be applied at point B to reach equilibrium.

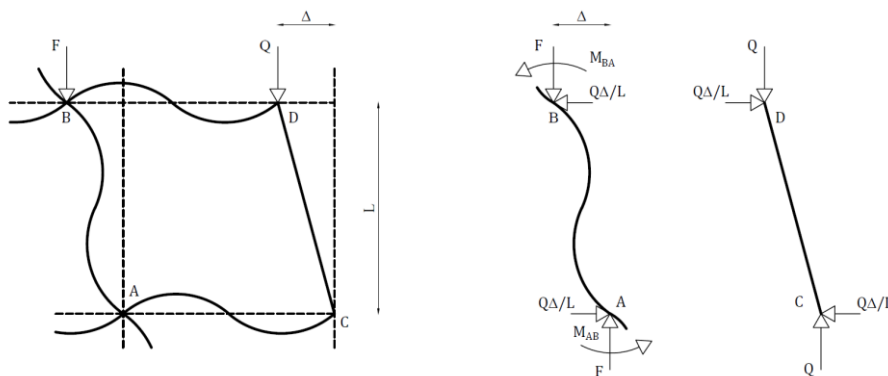


Figure 2: stabilizing column (A-B) and leaning column (C-D) [2]

Further steps for extending equation (3) are unknown, but G.L. Geschwindner (2002) [2] states the following: "Equations of equilibrium at the joints of column AB and the sway equilibrium equation can be written for the structure in the displaced configuration. Member end moment equations are then written using the slope deflection method, incorporating the stability functions necessary to account for the influence of axial load on column AB. Combining these equations and

setting the determinate of the coefficients equal to zero will yield the following buckling condition equation. (eq. 5)"

$$\frac{G_A G_B (\pi/K)^2 - 36}{6(G_A + G_B)} \left(1 + \frac{Q}{F}\right) - \frac{\frac{\pi}{K}}{\tan\left(\frac{\pi}{K}\right)} \left(1 + \frac{Q}{P}\right) + \frac{6 \tan\left(\frac{\pi}{2K}\right)}{(G_A + G_B) \left(\frac{\pi}{2K}\right)} \left(\frac{Q}{F}\right) + \left(\frac{Q}{F}\right) = 0 \quad (5)$$

If the load on the leaning column is zero, $Q = 0$, then equation (5) will reduce to equation (3). This suggests that equation (5) is correct.

1.3 YURA APPROACH

The Yura approach makes use of a kinematic model aiming for equilibrium. Consider the leaning column situation in a deformed frame as shown in Figure 3. Again, for an equilibrium of the leaning column CD, load Q must be balanced with a horizontal force with a magnitude of $Q\Delta/L$. When this horizontal force is applied in point B to reach equilibrium, one can see that in the deformed state the bending moment in point A will increase with $L \cdot \frac{Q\Delta}{L}$, resulting in a bending moment of $\Delta F + \Delta Q$. This indicates that column AB is loaded by $F+Q$. However, G.L. Geschwindner (2002) [2] states that this assumption that the buckling load equals $F+Q$ is slightly conservative since the buckled shape due to an axial load and the deflected shape due to a lateral load differ only slightly. This approach assumes that all stabilizing columns in a story buckle a side sway mode simultaneously.

To make this approach comparable with the other approaches, the result will be rewritten to critical load formulation with factor K_o :

$$F + Q = \frac{\pi^2 EI}{K_o^2 L_{cr}^2} \quad (6)$$

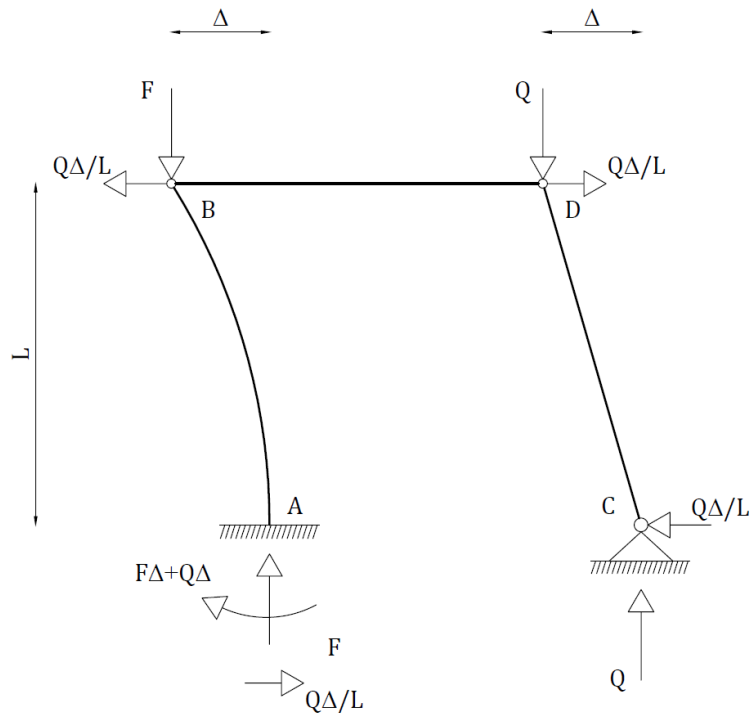


Figure 3: Equilibrium forces in stabilizing column (A-B) and leaning column (C-D) for Yura derivation

The leaning column effect is now taken into account in a new K_n -factor, given as:

$$F = \frac{\pi^2 EI}{K_n^2 L_{cr}^2} \quad (7)$$

Substituting $\frac{K_n^2}{K_o^2} = \frac{F+Q}{F}$ in (6) gives:

$$K_n = K_o \sqrt{\frac{F+Q}{F}} \quad (8)$$

Where K_o and K_n are modified effective length factors

1.4 LIM & MCNAMARA APPROACH

The approach of Lim & McNamara also assumes that all stabilizing columns in a story buckle a side sway mode simultaneously. These researchers developed the sway buckling equation through the use of stability functions and an eigenvalue solution [2]. The K_n -factor is given as:

$$K_n = K_o \sqrt{1 + n \frac{F_o}{F_n}} \quad (9)$$

Where K_o and K_n are modified effective length factors, $n = \sum Q / \sum F$, F_o is the eigenvalue solution for a frame without leaning columns and F_n is the eigenvalue solution for a frame with leaning columns. G.L. Geschwindner (2002) [2] states that normal end conditions lead to $F_o / F_n = 1$. Substituting this in equation (9) results in the same equation for K_n as the Yura approach:

$$K_n = K_o \sqrt{1 + \frac{\sum Q}{\sum F}} = K_o \sqrt{\frac{F+Q}{F}} \quad (10)$$

This suggests that the approach of Lim & McNamara also indicates that the stabilizing column of Figure 3 is loaded by $F+Q$.

1.5 LEMESSURIER APPROACH

The previous approach determined a constant value for a story by which the nomograph value of K_o was modified. This approach takes into account the contribution to the lateral resistance of each column along with the magnitude of the load on that column [2]. This is carried out by multiplying the constant value for a story with I/F , where I is the moment of inertia of the column and F the load of the stabilizing column. This gives the following equation for the effective length factor K_i :

$$K_n^2 = \frac{I_i \pi^2 \sum F + \sum Q + \sum (C_L \cdot F)}{\sum (\beta \cdot I)} \quad (11)$$

Where:

$$\beta = \frac{6(G_A + G_B) + 36}{2(G_A + G_B) + G_A \cdot G_B + 3} \quad (12)$$

$C_L = \frac{\beta \cdot K_o^2}{\pi^2} - 1$, but is equal to 0 for leaning columns;

K_n = effective length of the column, accounting for leaning columns;

F, Q = load on stabilizing column and leaning column respectively;

$\sum F$ and $\sum Q$ represent the total load on the columns in a story. $\sum(\beta \cdot I)$ represents the sum of $\beta \cdot I$ for each column participating in lateral sway resistance [2].

1.6 AMERICAN INSTITUTE OF STEEL CONSTRUCTION

Since the equation of LeMussurier is experienced as complex, the AISC modified these equations to a more simpler format. G.L. Geschwindner (2002) [2] states that for the story-buckling model it is assumed that there is no reduction in column stiffness due to the presence of axial load, leading to $C_L = 0$, which gives $\beta = \pi^2 / K_o^2$. Substituting this in (14) gives:

$$K_n = \sqrt{\frac{I \cdot \sum F + \sum Q}{F \cdot \sum(I/K_o^2)}} \quad (13)$$

G.L. Geschwindner (2002) [2] states that equation (16) is the same equation as presented in the LRFD. When a structure as shown in Figure 3 is considered, this equation again reduces to equation (11). That means that also this approach indicates that the stabilizing column of Figure 3 is loaded by $F+Q$.

2 LITERATURE REGARDING EC 1993-1-1

In 2012, the Eurocode was adopted in The Netherlands. The Eurocode is a European standard to assess the structural performance and safety of all possible building structures. The Eurocode EC 1993-1-1 is the basic part containing the design rules for steel structures. In Clause 6.3.3(4), the EC 1993-1-1 offers a three-dimensional design rule for members in bending and axial compression. For understanding the theory and the derivation of the three-dimensional design rule, it will be studied in the next sections. Besides that, the design rules, including Annex A and Annex B, will be translated into a two-dimensional format, which makes it comparable with the design rules from the NEN 6771.

The EC 1993-1-1 classifies steel cross-sections in different classes, namely cross-section classes 1, 2, 3, and 4. Here can classes 1 and 2 develop their full plastic moment resistance and class 3 only their elastic moment resistance. Class 4 only can develop an elastic moment resistance over an effective area, since it is subjected to the effects of local buckling. Since the effect of local buckling is not of interest, cross-sections of class 4 will not be considered in the upcoming sections. This is also the case for lateral-torsional buckling (LTB) because a two-dimensional case in this graduation project will be considered and any out-of-plane deformations in the Finite Element (FE) model will not be allowed.

2.1 MEMBERS IN COMPRESSION

Members in compression present the first term of the design rules for bending and axial compression. For that reason, this behavior will be discussed separately. The design rule in the EC 1993-1-1 for members in compression is given by the following:

$$\frac{N_{Ed}}{\chi \cdot A \cdot f_y} \leq \frac{1}{\gamma_1} \quad (14)$$

Or in the more simplistic format:

$$\frac{N_{Ed}}{(\chi \cdot N_{Rd})} \quad (15)$$

Equation (15) represents the member resistance unity-check due to axial loading of the column. Value N_{Rd} represents the plastic load resistance of the member and is given by $N_{Rd} = A \cdot f_y / \gamma_1$, where A is the area of the cross-section, f_y the yield strength of the material and γ_1 is the partial material factor. The reduction factor χ for flexural buckling can be derived from the design rules for members in only compression, given as the following:

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} \quad (16)$$

Where, for class 1, 2, and 3 cross-sections, $\bar{\lambda}$ is the relative slenderness. If $\bar{\lambda} < 0.2$, then the buckling effect may be ignored, and only cross-sectional checks apply [4]. $\bar{\lambda}$ is given by:

$$\bar{\lambda} = \sqrt{N_{Rk} / F_{cr}} \quad (17)$$

Where N_{Rk} is the characteristic axial resistance, given by $A \cdot f_y$ and F_{cr} is the elastic critical buckling load given by $F_{cr} = \pi^2 EI / L_{cr}^2$. The intermediate factor ϕ is given by:

$$\phi = 0.5[1 + \alpha(\bar{\lambda} - 0.2) + \bar{\lambda}^2] \quad (18)$$

The analytical formulation of the buckling curves was derived by Maquoi and Rondal [5]. This formulation was based on the Ayrton-Perry format.

$$\left(\frac{1}{\chi} - 1\right)(1 - \chi \cdot \bar{\lambda}^2) = \eta\chi = \alpha\chi(\bar{\lambda} - 0.2) \quad (19)$$

The value η represents the generalized initial imperfection. This value can estimate the buckling phenomenon of initial imperfections such as residual stresses, initial out-of-straightness, or eccentrically applied forces [5]. Value 0.2 in equation (18) represents the plateau in the buckling curves, where $\chi=1.0$, as can be seen in Figure 4. Value α in equation (18) is the imperfection factor and depends on the type of the cross-section, the thickness of the plates, the yield strength, and the buckling plane [5]. The value α can be obtained from Table 1, where the relating buckling curve can be obtained from Table 2. The buckling curves are shown in Figure 4.

Table 1: Imperfection factor α from the EC 1993-1-1

BUCKLING CURVE	a ₀	a	b	c	d
Imperfection factor α	0.13	0.21	0.34	0.49	0.76

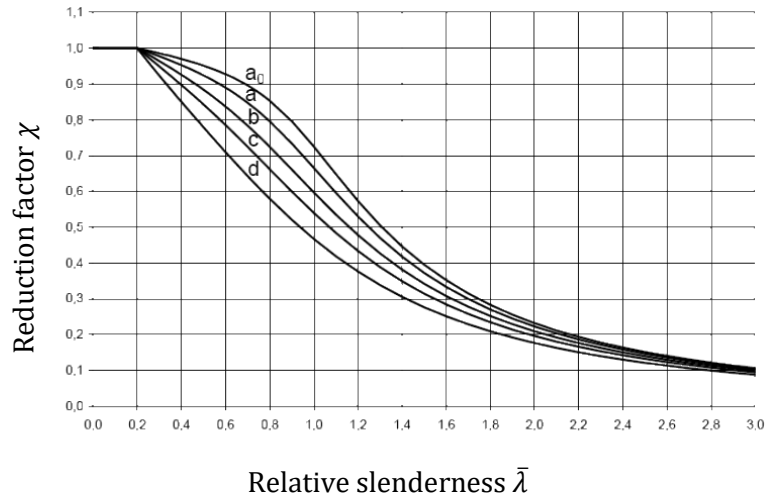
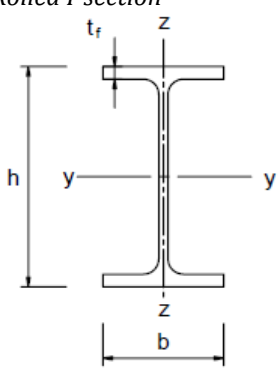


Figure 4: Buckling curves from the EC 1993-1-1 [6]

Table 2: Buckling curves for the relation cross-sections [4]

CROSS-SECTION	LIMITS	BUCKLING ABOUT AXIS	BUCKLING CURVE	
			S235 S275 S355 S420	S460
	$h/b > 1,2$ $t_f \leq 40 \text{ mm}$	y-y z-z	a	a ₀
	$40 \text{ mm} < t_f \leq 100 \text{ mm}$	y-y z-z	b c	a a
	$h/b \leq 1,2$ $t_f \leq 100 \text{ mm}$	y-y z-z	b c	a a
	$t_f > 100 \text{ mm}$	y-y z-z	d d	c c

2.2 MEMBERS IN BENDING AND AXIAL COMPRESSION

A member under bending and compression can have additional second-order moments due to second-order displacements, the so-called P-δ effect. Figure 5 shows a column with an imperfection e_0 , and a column subjected to a vertical and horizontal load. The moment equilibrium leads to the theory and abbreviations that are used to obtain the design rules in the NEN 6771, with the P-δ effect taken into account. To get an insight behind the theory of the EC 1993-1-1 the same column will be considered, but now with an initial bow imperfection defined by a transverse displacement e_0 , subjected to bending moment and axial compression. This is illustrated in Figure 5, which also shows the bending moment diagram due to the first-order moments and the second-order moments that result from the lateral deformation [5].

The present approach of EC3-1-1 is based on a linear-additive interaction formula, given by:

$$f\left(\frac{F}{F_u}, \frac{M_y}{M_{uy}}, \frac{M_z}{M_{uz}}\right) \leq 1.0 \quad (20)$$

Where F , M_y , and M_z are the applied forces, and F_u , M_{uy} , and M_{uz} are the design resistances. This interaction formula treats a three-dimensional case and takes into account the instability phenomena. Luís Simões da Silva et al. [5] (2010) state that: “The development of the design rules, and in particular those adopted by EC 1993-1-1, is quite complex, as they have to incorporate two instability modes, flexural buckling and lateral-torsional buckling (or a combination of both), different cross-sectional shapes and several shapes of bending moment diagram, among other aspects.” In the next sections, these aspects will be discussed separately, leading to the design rules that are presented in the EC 1993-1-1.

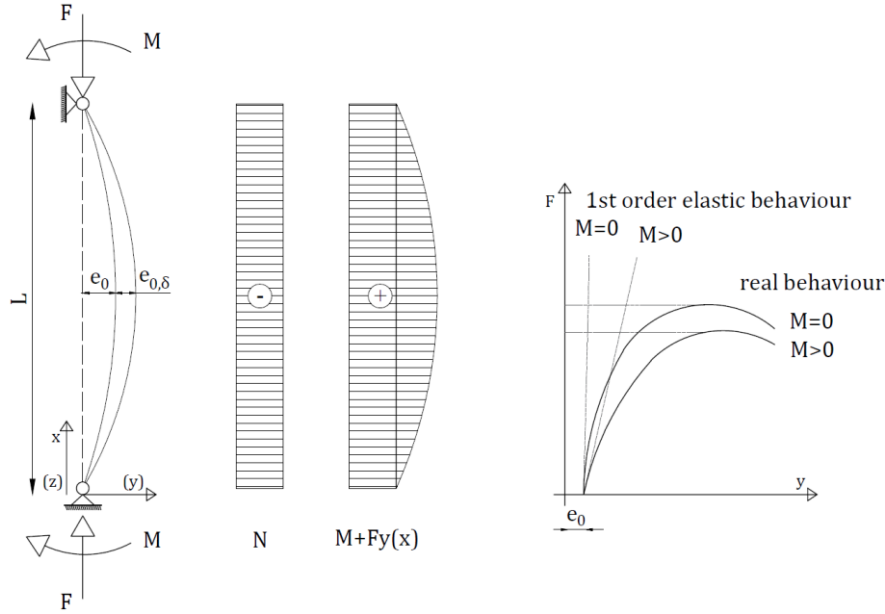


Figure 5: Behaviour of a member subjected to bending and compression [5]

2.2.1 Elastic flexural buckling

A member with an initial imperfection and loaded with axial compression N_{Ed} can be expressed by the following format: [4]

$$\frac{N_{Ed}}{N_{Rd}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{N_{Ed} \cdot e_0}{M_{Rd}} \leq 1 \quad (21)$$

The equivalent geometrical imperfection e_0 is given by: [4]

$$e_0 = \frac{(1 - \chi)(1 - \chi \bar{\lambda}^2)}{\chi} \cdot \frac{M_{el,Rd}}{N_{pl,Rd}} \quad (22)$$

Where χ and $\bar{\lambda}$ the reduction factor for flexural buckling and the relative slenderness respectively, as given in section 2.1. $M_{el,Rd}$ and $N_{pl,Rd}$ are the elastic moment resistance and the plastic axial load resistance of the member respectively. When an additional first-order bending moment is acting on the member, equation (21) can be extended, leading to the following equation [4]:

$$\frac{N_{Ed}}{N_{Rd}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{N_{Ed} \cdot e_0}{M_{Rd}} + \frac{M_{Ed,max}^I}{M_{Rd}} \leq 1 \quad (23)$$

Where $M_{Ed,max}^{II}$ is the maximum second-order bending moment. With different possible loading cases, it may appear that the maximum second-order bending moment, acts on a different location than the first-order bending moment or the bending moment due to imperfections. This principle is illustrated in Figure 6.

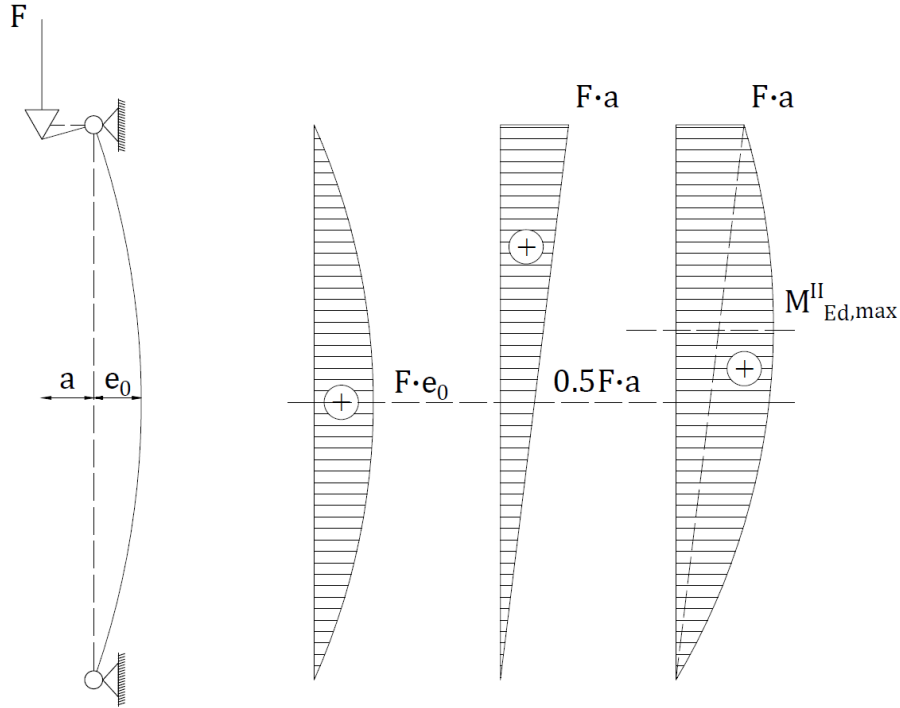


Figure 6: Column with an imperfection loaded on compression and a bending moment at one end [7]

The EC 1993-1-1 avoided the determination of the precise location of $M_{Ed,max}^{II}$, by using the so-called equivalent moment concepts. Boissonade et al. [4] (2010) describes this concept as the following: "This consists of replacing the actual first-order bending system on the member already subjected to the same axial force by a sinusoidal first-order bending moment that produces the same implied bending moment.". This leads to the term $C_m \cdot M_{Ed}$, where values for C_m will be discussed later. The maximum second-order bending moment $M_{Ed,max}^{II}$ can then be given as:

$$M_{Ed,max}^{II} = \frac{C_m \cdot M_{Ed}}{1 - \frac{N_{Ed}}{F_{cr}}} \quad (24)$$

With this term for the maximum second-order bending moment $M_{Ed,max}^{II}$, an elastic second-order check of the most loaded cross-section on the member can be derived from [4]:

$$\frac{N_{Ed}}{N_{Rd}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \cdot \frac{N_{Ed} \cdot e_0}{M_{Rd}} + \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \quad (25)$$

The EC 1993-1-1 has chosen to rewrite this formulation, for making the classical buckling term appear separately. This is given as the following:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + \mu \frac{1}{1 - \frac{N_{Ed}}{F_{cr}}} \frac{C_m \cdot M_{Ed}}{M_{Rd}} \leq 1 \quad \text{where } \mu = \frac{1 - \frac{N_{Ed}}{F_{cr}}}{1 - \chi \cdot \frac{N_{Ed}}{F_{cr}}} \quad (26)$$

2.2.2 Elastic-plastic flexural buckling without lateral torsional buckling

Equation (26) is based on elastic second-order theory, so are only valid for class 3 cross-sections. To allow yielding along the member, equation (26) is extended, given by [5]:

$$\frac{N_{Ed}}{\chi \cdot N_{pl,Rd}} + \mu \frac{1}{1 - \frac{N_{Ed}}{N_{cr}}} \cdot \frac{C_m \cdot M_{Ed}}{C \cdot M_{pl,Rd}} \quad (27)$$

In this case, the full plastic moment resistance presents bending moment resistance. Factor C takes into account the plasticity effects in the interaction between mono-axial bending and axial force along the member [4]. This will prevent that the full plastic bending resistance is taken into account, but an intermediate elastic-plastic value, given by $C \cdot M_{pl,Rd}$. This takes care of Class 1 and 2 cross-sections, but also still the class 3 cross-section.

To make a clear presentation for the design rules for members in bending and axial compression they minimized the design rule. The final presentation format of the design rules in the EC 1993-1-1 are the following, presenting three-dimensional design rules:

$$\frac{N_{Ed}}{\chi_y \cdot N_{Rk}} + k_{yy} \frac{M_{y,Ed} + \Delta M_{y,Ed}}{\chi_{LT} \frac{M_{y,Rk}}{\gamma_{m1}}} + k_{yz} \frac{M_{z,Ed} + \Delta M_{z,Ed}}{\chi_{LT} \frac{M_{z,Rk}}{\gamma_{m1}}} \leq 1 \quad (28)$$

$$\frac{N_{Ed}}{\chi_z \cdot N_{Rk}} + k_{zy} \frac{M_{y,Ed} + \Delta M_{y,Ed}}{\chi_{LT} \frac{M_{y,Rk}}{\gamma_{m1}}} + k_{zz} \frac{M_{z,Ed} + \Delta M_{z,Ed}}{\chi_{LT} \frac{M_{z,Rk}}{\gamma_{m1}}} \leq 1 \quad (29)$$

For comparison reasons with the NEN 6771, the design rules (28) and (29) for columns loaded in compression and three-dimensional bending is rewritten into a two-dimensional case and given as the following:

$$\frac{N_{Ed}}{\chi \cdot N_{Rd}} + k \frac{M_{Ed}}{M_{Rd}} \leq 1 \quad (30)$$

Where:

- N_{Ed} = design value of the load;
- N_{Rd} = design value of the plastic load resistance of the column cross-section;
- χ = buckling reduction factor;
- k = interaction factors depending on the relevant instability and plasticity phenomena [5];
- M_{Ed} = design value of the maximum bending moment along the member;
- M_{Rd} = elastic or plastic moment resistance of the column, depending on the cross-section class.

Equation (30) is valid for a column under compression and bending in a two-dimensional frame. Compared to equation (28) and (29), the three-dimensional values are converted to a two-dimensional case, and the lateral-torsional buckling reduction factor χ_{LT} and the bending moments due to the shift of the center of gravity ΔM are left out. Reduction factor χ_{LT} is left out because a two-dimensional case is considered and any out-of-plane (lateral torsional buckling) deformations in the FE-model will not be allowed. The reason for letting out bending moment ΔM is that profile classes 1, 2, and 3 will not be considered. According to Table 3, obtained from the EC 1993-1-1, this leads to $\Delta M = 0$.

Table 3: Values for $F_{Rk} = f_y \cdot A_i$, $M_{Rk} = f_y \cdot W_i$ and ΔM from the EC 1993-1-1

CLASS	1	2	3	4
A_i	A	A	A	A_{eff}
W	W_{pl}	W_{pl}	W_{el}	W_{eff}
ΔM_y	0	0	0	$e_{N,y\zeta} \cdot N_{Ed}$

The term $k \frac{M_{Ed}}{M_{Rd}}$ in equation (30) represents the member resistance unity-check due to a bending moment in the column. Value M_{Rd} is given by $M_{Rd} = W \cdot f_y / \gamma_1$, where γ_1 is the partial material factor and the section modulus W can be found in Table 3, where W_{pl} and W_{el} are the plastic and elastic section modulus respectively. Value k in equation (30) represents the interaction factors that depend on the relevant instability and plasticity phenomena [5]. The value k can be derived from two different methods. Back in the days when the EC 1993-1-1 was written, two different teams of researchers treated the field of beam-columns. This has led to two different methods that are both included in the EC 1993-1-1 as Annex A (method 1) and Annex B (method 2). Both methods consider the most complex behavior of a member subjected to compression and biaxial bending, including all possible interactions and non-linear effects [4]. Boissonade et al. (2006) [4] state that in method 1 all the influences of material and geometrical nonlinearities and interactions between loading components are reflected by separate factors. Boissonade et al. (2006) [4] also state that, in contrast, method 2 uses a reduced number of such factors as a result of globalization of several effects and calibration of the latter based on extensive numerical solutions. Therefore it is stated that Method 1 is better in identifying and accounting for structural effects, and Method 2 is better in focusing on the direct design of standard cases [4].

In the next paragraphs, only the design rules for a two-dimensional case will be discussed, since this is only of interest, where the EC 1993-1-1 offers the design rules for three-dimensional cases. This means that only the equations for the y-y plane are considered. Both Methods from Annex A and B for the derivation of factor K will be discussed in the next paragraphs.

2.2.2.1 Method 1

For Method 1, the following derivation of interaction factor k from Annex A is given:

Table 4: Interaction factor k for Method 1 from EC 1993-1-1 Annex A

INTERACTION FACTORS	ELASTIC SECTIONAL PROPERTIES (CLASS 3 OR 4)	PLASTIC SECTIONAL PROPERTIES (CLASS 1 OR 2)
k	$C_m C_{mLT} \frac{\mu_y}{1 - \frac{N_{Ed}}{F_{cr}}}$	$C_m C_{mLT} \frac{\mu_y}{1 - \frac{N_{Ed}}{F_{cr}}} \frac{1}{C_{yy}}$

According to Method 1, a member is not susceptible to torsional deformations if $I_T \geq I_y$, or when the following condition is verified:

$$\bar{\lambda}_0 \leq 0.2 \sqrt{C_1}^4 \sqrt{\left(1 - \frac{N_{Ed}}{N_{cr,z}}\right) \left(1 - \frac{N_{Ed}}{N_{cr,T}}\right)} \quad (31)$$

Since it is stated that lateral-torsional buckling is not considered, equation (31) is stated as true. This gives the following:

$$\text{If } \bar{\lambda}_0 \leq 0.2\sqrt{C_1} \cdot \sqrt[4]{\left(1 - \frac{N_{Ed}}{N_{cr,z}}\right)\left(1 - \frac{N_{Ed}}{N_{cr,T}}\right)} \text{ then: } C_m = C_{m,0}; C_{mLT} = 1.0 \quad (32)$$

According to Table 4, some additional terms for the derivation of k are given. The information includes the calculation of factor C_{yy} , this factor depends on the degree of plasticity in the cross-section at the collapse of the member [5].

$$\mu_y = \frac{1 - \frac{N_{Ed}}{F_{cr,y}}}{1 - \chi_y \frac{N_{Ed}}{F_{cr,y}}} \quad (33)$$

$$w_y = \frac{W_{pl,y}}{W_{el,y}} \leq 1.5 \quad (34)$$

$$n_{pl} = \frac{N_{Ed}}{N_{Rk}/\gamma_{M1}} \quad (35)$$

$$\alpha_{LT} = 1 - \frac{I_T}{I_y} \geq 0 \quad (36)$$

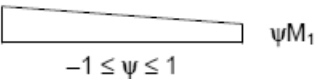
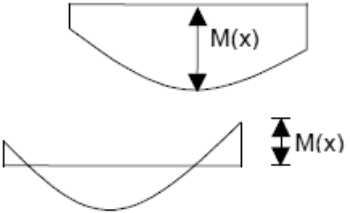
$$C_{yy} = 1 + (w_y - 1) \left[\left(2 - \frac{1.6}{w_y} C_{my}^2 \cdot \bar{\lambda}_{max} - \frac{1.6}{w_y} C_{my}^2 \cdot \bar{\lambda}_{max}^2 \right) n_{pl} - b_{LT} \right] \geq \frac{W_{el,y}}{W_{pl,y}} \quad (37)$$

$$b_{LT} = 0.5\alpha_{LT} \cdot \bar{\lambda}_0^2 \cdot \frac{M_{y,Ed}}{\chi_{LT} \cdot M_{pl,y,Rd}} \cdot \frac{M_{z,Ed}}{M_{pl,z,Rd}} \quad (38)$$

$$\bar{\lambda}_{max} = \max(\bar{\lambda}_y, \bar{\lambda}_z) \quad (39)$$

$\bar{\lambda}_0$ is the non-dimensional slenderness for lateral-torsional buckling due to uniform bending moment, that is, taking $\Psi_y = 1.0$ in Table 5. Based on the corresponding bending moment diagram, C_m should be derived from Table 5.

Table 5: Equivalent factors of uniform moment C_m

DIAGRAM OF MOMENTS		$C_{m,0}$
M_1		$C_{m,0} = 0.79 + 0.21\Psi_i + 0.36(\Psi_i - 0.33) \frac{N_{Ed}}{F_{cr,i}}$
		$C_{m,0} = 1 + \left(\frac{\pi^2 \cdot E \cdot I_i \cdot \delta_x }{L^2 \cdot M_{i,Ed}(x) } - 1 \right) \frac{N_{Ed}}{F_{cr,i}}$ $M_{i,Ed}(x)$ is the maximum moment $M_{y,Ed}$ or $M_{z,Ed}$ according to the first-order analyses $ \delta_x $ is the maximum lateral deflection δ_z (due to $M_{y,Ed}$) or δ_y (due to $M_{z,Ed}$) along the member

	$C_{m,0} = 1 - 0.18 \frac{N_{Ed}}{F_{cr,i}}$ $C_{m,0} = 1 + 0.03 \frac{N_{Ed}}{F_{cr,i}}$
---	---


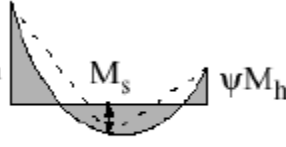
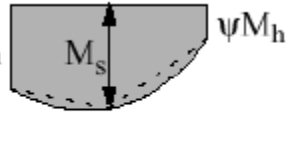
2.2.2.2 Method 2

Table 6 gives in the interaction factor k in members not susceptible to torsional deformations according to Method 2, Annex B. Table 7 gives the calculation of the equivalent uniform moment factors C_m .

Table 6: interaction factors k in members not susceptible to torsional deformations according to Method 2 [5]

INTERACTION FACTOR	TYPE OF SECTION	ELASTIC SECTIONAL PROPERTIES (CLASS 3 OR 4 SECTIONS)	PLASTIC SECTIONAL PROPERTIES (CLASS 1 OR 2 SECTIONS)
k_{yy}	I or H sections and rectangular hollow sections	$C_m \left(1 + 0.6 \bar{\lambda}_y \frac{N_{Ed}}{\chi_y \cdot N_{Rk} / \gamma_{M1}} \right)$ $\leq C_m \left(1 + 0.6 \frac{N_{Ed}}{\chi_y \cdot N_{Rk} / \gamma_{M1}} \right)$	$C_m \left(1 + (\bar{\lambda}_y - 0.2) \frac{N_{Ed}}{\chi_y \cdot N_{Rk} / \gamma_{M1}} \right)$ $\leq C_m \left(1 + 0.8 \frac{N_{Ed}}{\chi_y \cdot N_{Rk} / \gamma_{M1}} \right)$

Table 7: Equivalent factors of uniform moment C_m

DIAGRAM OF MOMENTS	RANGE		C_m	
			UNIFORM LOADING	CONCENTRATED LOAD
	$-1 \leq \Psi \leq 1$		$0.6 + 0.4\Psi \geq 0.4$	
 <p>$\alpha_s = M_s / M_h$</p>	$0 \leq \alpha_s \leq 1$	$-1 \leq \Psi \leq 1$	$0.2 + 0.8\alpha_s \geq 0.4$	$0.2 + 0.8\alpha_s \geq 0.4$
	$-1 \leq \alpha_s \leq 0$	$0 \leq \Psi \leq 1$	$0.1 - 0.8\alpha_s \geq 0.4$	$-0.8\alpha_s \geq 0.4$
		$-1 \leq \Psi \leq 0$	$0.1(1 - \Psi) - 0.8\alpha_s \geq 0.4$	$0.2(-\Psi) - 0.8\alpha_s \geq 0.4$
 <p>$\alpha_h = M_h / M_s$</p>	$-1 \leq \alpha_h \leq 1$	$-1 \leq \Psi \leq 1$	$0.95 + 0.05\alpha_h$	$0.90 + 0.10\alpha_h$
	$-1 \leq \alpha_h \leq 1$	$0 \leq \Psi \leq 1$	$0.95 + 0.05\alpha_h$	$0.90 + 0.10\alpha_h$
		$-1 \leq \Psi \leq 0$	$0.95 + 0.05\alpha_h(1 + 2\Psi)$	$0.90 + 0.10\alpha_h(1 + 2\Psi)$

In the calculation of α_s or α_h parameters, a hogging moment should be taken as negative and a sagging moment should be taken as positive.

For members with sway buckling mode, the equivalent uniform moment factor should be taken as $C_m = 0.9$

Factors C_m and C_{mLT} should be obtained from the diagram of bending moments between the relevant braced sections, according to the following:		
Moment factor C_m	Bending axis y-y	Points braced in direction z-z

For this graduation project, the statement "For members with sway buckling mode, the equivalent uniform moment factor should be taken as $C_m = 0.9$ " will be very important.

3 BIBLIOGRAPHY

- [1] Nederlands Normalisatie-instituut, "NEN 6770 - steel structures - basic requirements and basic rules for calculation of predominantly statically loaded structures," Delft, 1997.
- [2] L. F. Geschwindner, "A Practical Look at Frame Analysis, Stability and Leaning Columns," *Engineering Journal*, vol. Fourth Quarter, pp. 167-181, 2002.
- [3] American Institute of Steel Construction, Inc. (AISC), "Load and Resistance Factor Design Specification for Structural Steel Buildings," AISC, Chicago, 1999.
- [4] N. Boissonnade, R. Greiner, J. Jaspart and J. Lindner, Rules for Member Stability in EN 1993-1-1; Background documentation and design guidelines, Mem Martins, Portugal: ECCS, 2006.
- [5] L. Simões da Silva, R. Simões and H. Gervásio, "Eurocode 3: Design of Steel Structures; Part 1-1: General rules and rules for buildings," ECCS Eurocode Design Manuals, Mem Martins, Portugal, 2010.
- [6] Comité Européen de Normalisation, "Eurocode 3: Design of steel structures- Part 1-1: General rules and rules for buildings," CEN, Brussels, 2005.
- [7] H. Snijder and H. Steenbergen, Stabiteit 3; Staven belast op druk en buiging.

APPENDIX B – INPUT FEM MODEL PARAMETRIC STUDY

In the sheets below, the input is given for the parametric FE-model script. The sheets are subdivided in:

- Frames A, A-NL, A-V, A-H, A-NLH, and A-VH
- Frame B

id	Frame properties											Section properties										
	E	v	F	Q	H	$f_{y,d}$	f_u	ϵ	h	b1	b2	m	profile 1	h1	P1h	P1b	P1w	P1tf	I_y	$W_{y,d}$	$W_{y,pl}$	A
id	N/mm ²	[-]	N	N	N	N/mm ²	N/mm ²	%	mm	mm	mm	[-]	mm	mm	mm	mm	mm	mm	mm ³	mm ³	mm ³	mm ²
01	210000	0.3	1000000	1000000	0	355	490	0.06429	5000	5000	5000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
02	210000	0.3	1000000	1000000	0	355	490	0.06429	5000	7500	7500	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
03	210000	0.3	1000000	1000000	0	355	490	0.06429	5000	10000	10000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
A1	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	3000	3000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A2	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	4500	4500	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A3	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	6000	6000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A4	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	3000	3000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A5	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	4500	4500	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A6	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	6000	6000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A7	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	3000	3000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A8	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	4500	4500	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A9	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	6000	6000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A10	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	3000	3000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
A11	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	4500	4500	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
A12	210000	0.3	1000000	1000000	0	355	490	0.06429	3000	6000	6000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
01-NL	210000	0.3	1000000	0	0	355	490	0.06429	5000	5000	5000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
02-NL	210000	0.3	1000000	0	0	355	490	0.06429	5000	7500	7500	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
03-NL	210000	0.3	1000000	0	0	355	490	0.06429	5000	10000	10000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
A1-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	3000	3000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A2-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	4500	4500	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A3-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	6000	6000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A4-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	3000	3000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A5-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	4500	4500	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A6-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	6000	6000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A7-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	3000	3000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A8-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	4500	4500	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A9-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	6000	6000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A10-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	3000	3000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
A11-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	4500	4500	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
A12-NL	210000	0.3	1000000	0	0	355	490	0.06429	3000	6000	6000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
01-V	210000	0.3	1000000	10000000	0	355	490	0.06429	5000	5000	5000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
02-V	210000	0.3	1000000	10000000	0	355	490	0.06429	5000	7500	7500	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
03-V	210000	0.3	1000000	10000000	0	355	490	0.06429	5000	10000	10000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
A1-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	3000	3000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A2-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	4500	4500	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A3-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	6000	6000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A4-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	3000	3000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A5-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	4500	4500	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A6-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	6000	6000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A7-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	3000	3000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A8-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	4500	4500	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A9-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	6000	6000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156
A10-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	3000	3000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
A11-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	4500	4500	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
A12-V	210000	0.3	1000000	10000000	0	355	490	0.06429	3000	6000	6000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232
01-H	210000	0.3	1000000	1000000	12500	355	490	0.06429	5000	5000	5000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
02-H	210000	0.3	1000000	1000000	18750	355	490	0.06429	5000	7500	7500	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
03-H	210000	0.3	1000000	1000000	25000	355	490	0.06429	5000	10000	10000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627
A1-H	210000	0.3	1000000	1000000	4500	355	490	0.06429	3000	3000	3000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A2-H	210000	0.3	1000000	1000000	6750	355	490	0.06429	3000	4500	4500	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A3-H	210000	0.3	1000000	1000000	9000	355	490	0.06429	3000	6000	6000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410
A4-H	210000	0.3	1000000	1000000	4500	355	490	0.06429	3000	3000	3000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A5-H	210000	0.3	1000000	1000000	6750	355	490	0.06429	3000	4500	4500	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684
A6-H																						

id		Section properties											Model								
id	A	section class	profile 5	h5	P5h	P5b	P5w	P5tf	$I_{y,el}$	$W_{y,el}$	$W_{y,pl}$	A	section class	joint	condition	C	ma1	mb1	mc1	rs1	
id	mm ²	[-]		mm	mm	mm	mm	mm	mm ⁴	mm ³	mm ³	mm ²	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	
01	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1	78.96666667	32.75	25	0.5	
02	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1.5	78.96666667	32.75	25	0.5	
03	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	2	78.96666667	32.75	25	0.5	
A1	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1	1	48.36666667	12.25	10	0.5	
A2	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1.5	48.36666667	12.25	10	0.5		
A3	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	48.36666667	12.25	10	0.5		
A4	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1	1	47.76666667	16.75	13.33333333	0.5	
A5	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1.5	47.76666667	16.75	13.33333333	0.5		
A6	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	47.76666667	16.75	13.33333333	0.5		
A7	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1	1	46.86666667	23.5	18.33333333	0.5	
A8	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1.5	46.86666667	23.5	18.33333333	0.5		
A9	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	46.86666667	23.5	18.33333333	0.5		
A10	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1	1	45.93333333	30.5	23.33333333	0.5	
A11	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1.5	45.93333333	30.5	23.33333333	0.5		
A12	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	45.93333333	30.5	23.33333333	0.5		
id																					
01-NL	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1	78.96666667	32.75	25	0.5	
02-NL	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1.5	78.96666667	32.75	25	0.5	
03-NL	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	2	78.96666667	32.75	25	0.5	
A1-NL	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1	1	48.36666667	12.25	10	0.5	
A2-NL	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1.5	48.36666667	12.25	10	0.5		
A3-NL	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	48.36666667	12.25	10	0.5		
A4-NL	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1	1	47.76666667	16.75	13.33333333	0.5	
A5-NL	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1.5	47.76666667	16.75	13.33333333	0.5		
A6-NL	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	47.76666667	16.75	13.33333333	0.5		
A7-NL	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1	1	46.86666667	23.5	18.33333333	0.5	
A8-NL	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1.5	46.86666667	23.5	18.33333333	0.5		
A9-NL	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	46.86666667	23.5	18.33333333	0.5		
A10-NL	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1	1	45.93333333	30.5	23.33333333	0.5	
A11-NL	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1.5	45.93333333	30.5	23.33333333	0.5		
A12-NL	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	45.93333333	30.5	23.33333333	0.5		
id																					
01-V	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1	78.96666667	32.75	25	0.5	
02-V	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1.5	78.96666667	32.75	25	0.5	
03-V	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	2	78.96666667	32.75	25	0.5	
A1-V	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1	1	48.36666667	12.25	10	0.5	
A2-V	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1.5	48.36666667	12.25	10	0.5		
A3-V	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	48.36666667	12.25	10	0.5		
A4-V	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1	1	47.76666667	16.75	13.33333333	0.5	
A5-V	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1.5	47.76666667	16.75	13.33333333	0.5		
A6-V	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	47.76666667	16.75	13.33333333	0.5		
A7-V	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1	1	46.86666667	23.5	18.33333333	0.5	
A8-V	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1.5	46.86666667	23.5	18.33333333	0.5		
A9-V	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	46.86666667	23.5	18.33333333	0.5		
A10-V	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1	1	45.93333333	30.5	23.33333333	0.5	
A11-V	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1.5	45.93333333	30.5	23.33333333	0.5		
A12-V	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	45.93333333	30.5	23.33333333	0.5		
id																					
01-H	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1	78.96666667	32.75	25	0.5	
02-H	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	1.5	78.96666667	32.75	25	0.5	
03-H	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	1	2	78.96666667	32.75	25	0.5	
A1-H	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1	1	48.36666667	12.25	10	0.5	
A2-H	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	1.5	48.36666667	12.25	10	0.5		
A3-H	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	48.36666667	12.25	10	0.5		
A4-H	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1	1	47.76666667	16.75	13.33333333	0.5	
A5-H	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	1.5	47.76666667	16.75	13.33333333	0.5		
A6-H	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	47.76666667	16.75	13.33333333	0.5		
A7-H	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1	1	46.86666667	23.5	18.33333333	0.5	
A8-H	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	1.5	46.86666667	23.5	18.33333333	0.5		
A9-H	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	46.86666667	23.5	18.33333333	0.5		
A10-H	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1	1	45.93333333	30.5	23.33333333	0.5	
A11-H	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	1.5	45.93333333	30.5	23.33333333	0.5		
A12-H	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	45.93333333	30.5	23.33333333	0.5		
id																					
01-VH	1062																				

Model													LBA				
id	ma2	mb2	mc2	rs2	ma4	mb4	mc4	rs4	ma5	mb5	mc5	rs5	α_{cr}	mode 1	mode 2	mode 3	F_{cr}
id	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	N
01	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	1.7898	1	64	335	1.79E+06
02	120.6333333	32.75	25	0.5	78.96666667	32.75	25	0.5	120.6333333	32.75	25	0.5	1.5768	1	40	330	1.58E+06
03	162.3	32.75	25	0.5	78.96666667	32.75	25	0.5	162.3	32.75	25	0.5	1.401	1	26	327	1.40E+06
A1	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	0.16897	1	4	7	1.69E+05
A2	73.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	73.36666667	12.25	10	0.5	0.14785	1	4	7	1.48E+05
A3	98.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	98.36666667	12.25	10	0.5	0.13099	1	4	7	1.31E+05
A4	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	0.46194	1	4	189	4.62E+05
A5	72.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	72.76666667	16.75	13.33333333	0.5	0.4056	1	4	189	4.06E+05
A6	97.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	97.76666667	16.75	13.33333333	0.5	0.35985	1	4	179	3.60E+05
A7	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	1.4712	1	134	140	1.47E+06
A8	71.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	71.86666667	23.5	18.33333333	0.5	1.3025	1	108	309	1.30E+06
A9	96.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	96.86666667	23.5	18.33333333	0.5	1.1601	1	72	74	1.16E+06
A10	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	3.5822	1	1	1	3.58E+06
A11	70.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	70.93333333	30.5	23.33333333	0.5	3.2065	1	1	1	3.21E+06
A12	95.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	95.93333333	30.5	23.33333333	0.5	2.8712	1	1	1	2.87E+06
01-NL	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	2.601	1	2	43	2.60E+06
02-NL	120.6333333	32.75	25	0.5	78.96666667	32.75	25	0.5	120.6333333	32.75	25	0.5	2.3089	1	2	35	2.31E+06
03-NL	162.3	32.75	25	0.5	78.96666667	32.75	25	0.5	162.3	32.75	25	0.5	2.0629	1	2	19	2.06E+06
A1-NL	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	0.24542	1	2	3	2.45E+05
A2-NL	73.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	73.36666667	12.25	10	0.5	0.21647	1	2	3	2.16E+05
A3-NL	98.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	98.36666667	12.25	10	0.5	0.19285	1	2	3	1.93E+05
A4-NL	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	0.67112	1	2	3	6.71E+05
A5-NL	72.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	72.76666667	16.75	13.33333333	0.5	0.59388	1	2	3	5.94E+05
A6-NL	97.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	97.76666667	16.75	13.33333333	0.5	0.52982	1	2	3	5.30E+05
A7-NL	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	2.1392	1	34	95	2.14E+06
A8-NL	71.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	71.86666667	23.5	18.33333333	0.5	1.9079	1	26	75	1.91E+06
A9-NL	96.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	96.86666667	23.5	18.33333333	0.5	1.7085	1	18	55	1.71E+06
A10-NL	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	5.2148	1	136	1	5.21E+06
A11-NL	70.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	70.93333333	30.5	23.33333333	0.5	4.6994	1	122	1	4.70E+06
A12-NL	95.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	95.93333333	30.5	23.33333333	0.5	4.2304	1	122	1	4.23E+06
01-V	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	0.46754	1	52	1	4.68E+05
02-V	120.6333333	32.75	25	0.5	78.96666667	32.75	25	0.5	120.6333333	32.75	25	0.5	0.40714	1	37	1	4.07E+05
03-V	162.3	32.75	25	0.5	78.96666667	32.75	25	0.5	162.3	32.75	25	0.5	0.35879	1	4	1	3.59E+05
A1-V	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	0.044217	1	4	7	4.42E+04
A2-V	73.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	73.36666667	12.25	10	0.5	0.038233	1	4	7	3.82E+04
A3-V	98.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	98.36666667	12.25	10	0.5	0.033598	1	4	7	3.36E+04
A4-V	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	0.12078	1	4	1	1.21E+05
A5-V	72.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	72.76666667	16.75	13.33333333	0.5	0.1048	1	4	1	1.05E+05
A6-V	97.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	97.76666667	16.75	13.33333333	0.5	0.09222	1	4	1	9.22E+04
A7-V	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	0.38371	1	1	1	3.84E+05
A8-V	71.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	71.86666667	23.5	18.33333333	0.5	0.3359	1	1	1	3.36E+05
A9-V	96.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	96.86666667	23.5	18.33333333	0.5	0.29672	1	1	1	2.97E+05
A10-V	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	0.93133	1	1	1	9.31E+05
A11-V	70.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	70.93333333	30.5	23.33333333	0.5	0.8248	1	1	1	8.25E+05
A12-V	95.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	95.93333333	30.5	23.33333333	0.5	0.73248	1	1	1	7.32E+05
01-H	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	1.7898	1	64	335	1.79E+06
02-H	120.6333333	32.75	25	0.5	78.96666667	32.75	25	0.5	120.6333333	32.75	25	0.5	1.5768	1	40	330	1.58E+06
03-H	162.3	32.75	25	0.5	78.96666667	32.75	25	0.5	162.3	32.75	25	0.5	1.401	1	26	327	1.40E+06
A1-H	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	0.16897	1	4	7	1.69E+05
A2-H	73.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	73.36666667	12.25	10	0.5	0.14785	1	4	7	1.48E+05
A3-H	98.36666667	12.25	10	0.5	48.36666667	12.25	10	0.5	98.36666667	12.25	10	0.5	0.13099	1	4	7	1.31E+05
A4-H	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	0.46194	1	4	189	4.62E+05
A5-H	72.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	72.76666667	16.75	13.33333333	0.5	0.4056	1	4	189	4.06E+05
A6-H	97.76666667	16.75	13.33333333	0.5	47.76666667	16.75	13.33333333	0.5	97.76666667	16.75	13.33333333	0.5	0.35985	1	4	179	3.60E+05
A7-H	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	1.4712	1	134	140	1.47E+06
A8-H	71.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	71.86666667	23.5	18.33333333	0.5	1.3025	1	108	309	1.30E+06
A9-H	96.86666667	23.5	18.33333333	0.5	46.86666667	23.5	18.33333333	0.5	96.86666667	23.5	18.33333333	0.5	1.1601	1	72	74	1.16E+06
A10-H	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	3.5822	1	1	1	3.58E+06
A11-H	70.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	70.93333333	30.5	23.33333333	0.5	3.2065	1	1	1	3.21E+06
A12-H	95.93333333	30.5	23.33333333	0.5	45.93333333	30.5	23.33333333	0.5	95.93333333	30.5	23.33333333	0.5	2.8712	1	1	1	2.87E+06
01-VH	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	0.46754	1	52	1	4.68E+05
02-VH	120.6333333	32.75	25	0.5	78.96666667	32.75	25	0.5	120.6333333	32.75	25	0.5	0.40714	1			

GMNIA III												
$P^2+C1*P+C2$ $P+(L_{cr}/10$ $=r^2-P^2$ $00)$ $Lcr/2-h$ $\sqrt{(r^2-x^2)}$ $y-P$												
id	$L_{cr} = \pi \sqrt{EI/(\alpha_c * F)}$	$L_{cr}/1000$	r^2-P^2	C_1	C_2	P	r	x	y	Δ	id	
id	mm	mm	mm	[-]	[-]	mm	mm	mm	mm	mm	id	
01	14147.7	14.15	50039600.6	28.30	200.16	1768459.8	1768474	2073.9	1768472.7	-12.93	id	
02	15073.0	15.07	56799135.6	30.15	227.20	1884122.5	1884138	2536.5	1884135.9	-13.37	id	
03	15990.8	15.99	63926393.3	31.98	255.71	1998841.6	1998858	2995.4	1998855.3	-13.75	id	
A1	8431.5	8.43	17772727.9	16.86	71.09	1053938.6	1053947	1215.8	1053946.3	-7.73	id	
A2	9013.7	9.01	20311517.4	18.03	81.25	1126703.0	1126712	1506.8	1126711.0	-8.01	id	
A3	9576.2	9.58	22925855.7	19.15	91.70	1197019.0	1197029	1788.1	1197027.3	-8.24	id	
A4	8458.4	8.46	17886239.0	16.92	71.54	1057298.9	1057307	1229.2	1057306.7	-7.74	id	
A5	9026.8	9.03	20370732.8	18.05	81.48	1128344.2	1128353	1513.4	1128352.2	-8.01	id	
A6	9583.4	9.58	2296592.6	19.17	91.84	1197925.5	1197935	1791.7	1197933.8	-8.24	id	
A7	8546.0	8.55	18258692.9	17.09	73.03	1068250.5	1068259	1273.0	1068258.3	-7.79	id	
A8	9082.6	9.08	20623561.6	18.17	82.49	1135324.7	1135334	1541.3	1135332.8	-8.04	id	
A9	9623.9	9.62	23155063.3	19.25	92.62	1202987.9	1202998	1812.0	1202996.2	-8.26	id	
A10	8672.6	8.67	18803555.4	17.35	75.21	1084072.3	1084081	1336.3	1084080.2	-7.85	id	
A11	9166.6	9.17	21006735.1	18.33	84.03	1145823.0	1145832	1583.3	1145831.1	-8.07	id	
A12	9687.1	9.69	23459910.9	19.37	93.84	1210881.0	1210891	1843.5	1210889.2	-8.28	id	
id												
01-NL	11736.0	11.74	34433247.6	23.47	137.73	1466990.4	1467002	868.0	1467001.9	-11.48	id	
02-NL	12456.2	12.46	38789413.6	24.91	155.16	1557022.5	1557035	1228.1	1557034.4	-11.97	id	
03-NL	13178.0	13.18	43415035.7	26.36	173.66	1647245.6	1647259	1589.0	1647258.0	-12.41	id	
A1-NL	6996.1	7.00	12236402.3	13.99	48.95	874510.7	874518	498.1	874517.6	-6.85	id	
A2-NL	7449.3	7.45	13872859.2	14.90	55.49	931153.5	931161	724.6	931160.6	-7.17	id	
A3-NL	7892.3	7.89	15571987.8	15.78	62.29	986530.0	986538	946.1	986537.4	-7.44	id	
A4-NL	7017.5	7.02	12311314.3	14.03	49.25	877183.6	877191	508.7	877190.4	-6.87	id	
A5-NL	7459.9	7.46	13912523.1	14.92	55.65	932483.6	932491	729.9	932489.8	-7.17	id	
A6-NL	7898.0	7.90	15594672.2	15.80	62.38	987248.3	987256	949.0	987255.7	-7.44	id	
A7-NL	7087.2	7.09	12557119.0	14.17	50.23	885897.1	885904	543.6	885904.0	-6.92	id	
A8-NL	7504.5	7.50	14079453.3	15.01	56.32	938061.2	938069	752.3	938068.4	-7.20	id	
A9-NL	7930.4	7.93	15722674.2	15.86	62.89	991291.7	991300	965.2	991299.2	-7.46	id	
A10-NL	7188.0	7.19	12916717.1	14.38	51.67	898492.3	898499	594.0	898499.3	-6.99	id	
A11-NL	7571.9	7.57	14333339.6	15.14	57.33	946481.1	946489	785.9	946488.4	-7.25	id	
A12-NL	7980.6	7.98	15922394.2	15.96	63.69	997567.9	997576	990.3	997575.4	-7.49	id	
id												
01-V	27680.9	27.68	191557678.6	55.36	766.23	3460095.2	3460123	8840.4	3460111.6	-16.39	id	
02-V	29663.2	29.66	219975627.7	59.33	879.90	3707879.0	3707909	9831.6	3707895.6	-16.63	id	
03-V	31598.7	31.60	249619212.0	63.20	998.48	3949819.7	3949851	#####	3949836.6	-16.84	id	
A1-V	16482.2	16.48	67915749.0	32.96	271.66	2060267.1	2060284	5241.1	2060276.9	-9.82	id	
A2-V	17725.3	17.73	78546841.9	35.45	314.19	2215657.5	2215675	5862.7	2215667.5	-9.97	id	
A3-V	18908.5	18.91	89382839.9	37.82	357.53	2363553.0	2363572	6454.2	2363563.1	-10.10	id	
A4-V	16541.9	16.54	68408422.2	33.08	273.63	2067726.3	2067743	5270.9	2067736.2	-9.82	id	
A5-V	17758.3	17.76	78839401.0	35.52	315.36	2219780.0	2219798	5879.2	2219789.9	-9.97	id	
A6-V	18930.8	18.93	89593920.1	37.86	358.38	2366342.2	2366361	6465.4	2366352.3	-10.10	id	
A7-V	16734.0	16.73	70006486.5	33.47	280.03	2091738.6	2091755	5367.0	2091748.5	-9.85	id	
A8-V	17885.3	17.89	79970791.8	35.77	319.88	2235650.8	2235669	5942.6	2235660.8	-9.99	id	
A9-V	19029.5	19.03	90530429.2	38.06	362.12	2378677.5	2378697	6514.7	2378687.6	-10.11	id	
A10-V	17008.8	17.01	72324628.5	34.02	289.30	2126088.7	2126106	5504.4	2126098.6	-9.88	id	
A11-V	18073.8	18.07	81665975.1	36.15	326.66	2259221.7	2259240	6036.9	2259231.7	-10.01	id	
A12-V	19179.0	19.18	91958956.2	38.36	367.84	2397371.2	2397390	6589.5	2397381.3	-10.12	id	
id												
01-H	14147.7	14.15	50039600.6	28.30	200.16	1768459.8	1768474	2073.9	1768472.7	-12.93	id	
02-H	15073.0	15.07	56799135.6	30.15	227.20	1884122.5	1884138	2536.5	1884135.9	-13.37	id	
03-H	15990.8	15.99	63926393.3	31.98	255.71	1998841.6	1998858	2995.4	1998855.3	-13.75	id	
A1-H	8431.5	8.43	17772727.9	16.86	71.09	1053938.6	1053947	1215.8	1053946.3	-7.73	id	
A2-H	9013.7	9.01	20311517.4	18.03	81.25	1126703.0	1126712	1506.8	1126711.0	-8.01	id	
A3-H	9576.2	9.58	22925855.7	19.15	91.70	1197019.0	1197029	1788.1	1197027.3	-8.24	id	
A4-H	8458.4	8.46	17886239.0	16.92	71.54	1057298.9	1057307	1229.2	1057306.7	-7.74	id	
A5-H	9026.8	9.03	20370732.8	18.05	81.48	1128344.2	1128353	1513.4	1128352.2	-8.01	id	
A6-H	9583.4	9.58	2296592.6	19.17	91.84	1197925.5	1197935	1791.7	1197933.8	-8.24	id	
A7-H	8546.0	8.55	18258692.9	17.09	73.03	1068250.5	1068259	1273.0	1068258.3	-7.79	id	
A8-H	9082.6	9.08	20623561.6	18.17	82.49	1135324.7	1135334	1541.3	1135332.8	-8.04	id	
A9-H	9623.9	9.62	23155063.3	19.25	92.62	1202987.9	1202998	1812.0	1202996.2	-8.26	id	
A10-H	8672.6	8.67	18803555.4	17.35	75.21	1084072.3	1084081	1336.3	1084080.2	-7.85	id	
A11-H	9166.6	9.17	21006735.1	18.33	84.03	1145823.0	1145832	1583.3	1145831.1	-8.07	id	
A12-H	9687.1	9.69	23459910.9	19.37	93.84	1210881.0	1210891	1843.5	1210889.2	-8.28	id	
id												
01-VH	27680.9	27.68	191557678.6	55.36	766.23	3460095.2	3460123	8840.4	3460111.6	-16.39	id	
02-VH	29663.2	29.66	219975627.7	59.33	879.90	3707879.0	3707909	9831.6	3707895.6	-16.63	id	
03-VH	31598.7	31.60	249619212.0	63.20	998.48	3949819.7	3949851	#####	3949836.6	-16.84	id	
A1-VH	16482.2	16.48	67915749.0	32.96	271.66	2060267.1	2060284	5241.1	2060276.9	-9.82	id	
A2-VH	17725.3	17.73	78546841.9	35.45	314.19	2215657.5	2215675	5862.7	2215667.5	-9.97	id	
A3-VH	18908.5	18.91	89382839.9	37.82	357.53	2363553.0	2363572	6454.2	2363563.1	-10.10	id	
A4-VH	16541.9	16.54	68408422.2	33.08	273.63	2067726.3	2067743	5270.9	2067736.2	-9.82	id	
A5-VH	17758.3	17.76	78839401.0	35.52	315.36	2219780.0	2219798	5879.2	2219789.9	-9.97	id	
A6-VH	18930.8	18.93	89593920.1	37.86	358.38	2366342.2	2366361	6465.4	2366352.3	-10.10	id	
A7-VH	16734.0	16.73	70006486.5	33.47	280.03	2091738.6	2091755	5367.0	2091748.5	-9.85	id	
A8-VH	17885.3	17.89	79970791.8	35.77	319.88	2235650.8	2235669	5942.6	2235660.8	-9.99	id	
A9-VH	19029.5	19.03	90530429.2	38.06	362.12	2378677.5	2378697	6514.7	2378687.6	-10.11	id	
A10-VH	17008.8	17.01	72324628.5	34.02	289.30	2126088.7	2126106	5504.4	2126098.6	-9.88	id	
A11-VH	18073.8	18.07	81665975.1	36.15	326.66	2259221.7	2259240	6036.9	2259231.7	-10.01	id	
A12-VH	19179.0	19.18	91958956.2	38.36	367.84	2397371.2	2397390	6589.5	2397381.3	-10.12	id	
id												
01-NLH	11736.0	11.74	34433247.6	23.47	137.73	1466990.4	1467002	868.0	1467001.9	-11.48	id	
02-NLH	12456.2	12.46	38789413.6	24.91	155.16	1557022.5	1557035	1228.1	1557034.4	-11.97	id	
03-NLH	13178.0	13.18	43415035.7	26.36	173.66	1647245.6	1647259	1589.0	1647258.0	-12.41	id	
A1-NLH	6996.1	7.00	12236402.3	13.99	48.95	874510.7	874518	498.1	874517.6	-6.85	id	
A2-NLH	7449.3	7.45	13872859.2	14.90	55.49	931153.5	931161	724.6	931160.6	-7.17	id	
A3-NLH	7892.3	7.89	15571987.8	15.78	62.29	986530.0	986538	946.1	986537.4	-7.44	id	
A4-NLH	7017.5	7.02	12311314.3	14.03	49.25	877183.6	877191	508.7	877190.4	-6.87	id	
A5-NLH	7459.9	7.46	13912523.1	14.92	55.65	932483.6	932491	729.9	932489.8	-7.17	id	
A6-NLH	7898.0	7.90	15594672.2	15.80	62.38	987248.3	987256	949.0	987255.7	-7.44	id	
A7-NLH	7087.2	7.09	12557119.0	14.17	50.23	885897.1	885904	543.6	885904.0	-6.92	id	
A8-NLH	7504.5	7.50	14079453.3	15.01	56.32	938061.2	938069	752.3	938068.4	-7.2		

Frame properties												Section properties											
id	E	v	F	Q	N	f_{td}	f_t	ϵ	h	b1	b2	m	profile 1	h1	P1h	P1b	P1w	P1f	I_y	$W_{y,el}$	$W_{y,pl}$	A	section class
id	N/mm ²	[-]	N	N	N/mm ²	N/mm ²	N/mm ²	%	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm ⁴	mm ³	mm ³	mm ²	[-]
B01	210000	0.3	1000000	1000000	355	490	0.06429	5000	5000	5000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	
B02	210000	0.3	1000000	1000000	355	490	0.06429	5000	7500	7500	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	
B03	210000	0.3	1000000	1000000	355	490	0.06429	5000	10000	10000	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	
B1	210000	0.3	1000000	1000000	355	490	0.06429	3000	3000	3000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	
B2	210000	0.3	1000000	1000000	355	490	0.06429	3000	4500	4500	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	
B3	210000	0.3	1000000	1000000	355	490	0.06429	3000	6000	6000	3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	
B4	210000	0.3	1000000	1000000	355	490	0.06429	3000	3000	3000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	
B5	210000	0.3	1000000	1000000	355	490	0.06429	3000	4500	4500	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	
B6	210000	0.3	1000000	1000000	355	490	0.06429	3000	6000	6000	3	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	
B7	210000	0.3	1000000	1000000	355	490	0.06429	3000	3000	3000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	
B8	210000	0.3	1000000	1000000	355	490	0.06429	3000	4500	4500	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	
B9	210000	0.3	1000000	1000000	355	490	0.06429	3000	6000	6000	3	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	
B10	210000	0.3	1000000	1000000	355	490	0.06429	3000	3000	3000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	
B11	210000	0.3	1000000	1000000	355	490	0.06429	3000	4500	4500	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	
B12	210000	0.3	1000000	1000000	355	490	0.06429	3000	6000	6000	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	
id	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23

Section properties																							
id	profile 2	h2	P2h	P2b	P2w	P2f	I_y	$W_{y,el}$	$W_{y,pl}$	A	section class	profile 4	h4	P4h	P4b	P4w	P4f	I_y	$W_{y,el}$	$W_{y,pl}$	A	section class	
id	mm	mm	mm	mm	mm	mm	mm ⁴	mm ³	mm ³	mm ²	[-]	mm	mm	mm	mm	mm	mm	mm ⁴	mm ³	mm ³	mm ²	[-]	
B01	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	
B02	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	
B03	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	
B1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	
B2	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	
B3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	
B4	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	
B5	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	
B6	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	
B7	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	
B8	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	
B9	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	
B10	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	
B11	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	
B12	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	
id	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	

Section properties												Model										
id	profile 5	h5	P5h	P5b	P5w	P5f	I_y	$W_{y,el}$	$W_{y,pl}$	A	section class	joint	condition	C	ma1	mb1	mc1	rs1	ma2			
id	mm	mm	mm	mm	mm	mm	mm ⁴	mm ³	mm ³	mm ²	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]			
B01	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	2	1	78.96666667	32.75	25	0.5	78.96666667			
B02	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	2	1.5	78.96666667	32.75	25	0.5	120.6333333			
B03	HEA300	290	262	300	8.5	14	1.7285E+08	1.1920E+06	1.3051E+06	10627	3	1	2	2	78.96666667	32.75	25	0.5	162.3			
B1	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	1	48.36666667	12.25	10	0.5	48.36666667			
B2	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	1.5	48.36666667	12.25	10	0.5	73.36666667			
B3	HEA120	114	98	120	5	8	5.7957E+06	1.0168E+05	1.1377E+05	2410	1	1	2	2	48.36666667	12.25	10	0.5	98.36666667			
B4	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	1	47.76666667	16.75	13.33333333	0.5	47.76666667			
B5	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	1.5	47.76666667	16.75	13.33333333	0.5	72.76666667			
B6	HEA160	152	134	160	6	9	1.5946E+07	2.0981E+05	2.3285E+05	3684	1	1	2	2	47.76666667	16.75	13.33333333	0.5	97.76666667			
B7	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	1	46.86666667	23.5	18.33333333	0.5	46.86666667			
B8	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	1.5	46.86666667	23.5	18.33333333	0.5	71.86666667			
B9	HEA220	210	188	220	7	11	5.1842E+07	4.9373E+05	5.4343E+05	6156	2	1	2	2	46.86666667	23.5	18.33333333	0.5	96.86666667			
B10	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	1	45.93333333	30.5	23.33333333	0.5	45.93333333			
B11	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	1.5	45.93333333	30.5	23.33333333	0.5	70.93333333			
B12	HEA280	270	244	280	8	13	1.3000E+08	9.6294E+05	1.0546E+06	9232	3	1	2	2	45.93333333	30.5	23.33333333	0.5	95.93333333			
id	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65		

Model												LBA										
id	mb2	mc2	rs2	ma4	mb4	mc4	rs4	ma5	mb5	mc5	rs5	α_{cr}	mode 1	mode 2	mode 3	F_{cr}						
id	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	N						
B01	32.75	25	0.5	78.96666667	32.75	25	0.5	78.96666667	32.75	25	0.5	7.0456	1	2	1	7.05E+06						
B02	32.75	25	0.5	78.96666667	32.75	25	0.5	120.6333333	32.75	25	0.5	6.3849	1	2	1							

APPENDIX C – INPUT AND OUTPUT FOR DESIGN RULES

In the sheets below, the in- and output of the design rules are given. The sheets are sub-divided in:

- Frames A, A-NL, A-V, A-H, A-NLH, and A-VH – NEN 6771 with nomogram
- Frames A, A-NL, A-V, A-H, A-NLH, and A-VH – Modified EN 1993-1-1 with nomogram
- Frames A, A-NL, A-V, A-H, A-NLH, and A-VH – Current EN 1993-1-1 with LBA, Yura, and Geschwindner
- Frame B – NEN 6771 with nomogram
- Frame B – Modified EN 1993-1-1 with nomogram
- Frame B – Current EN 1993-1-1 with LBA, Yura, and Geschwindner

NEN 6771 with Nomogram															
variable for solve function			$(E I_x / L_x) / (\mu (E I_b / L_b))$		$C_A \cdot C_B \cdot \lambda^2 \cdot \sin(\lambda)$	$(C_A + C_B) \cdot \lambda^2 \cdot \cos(\lambda) + \sin(\lambda)$	eq1-eq2=0	$\pi / \lambda (K_0)$	$\pi / \lambda \cdot h$	$\sqrt{(I/A)}$	$\pi^* \sqrt{(E / f_{yd})}$	L_{cr} / i	λ / λ_e	$W_{pl,yd}$	
N_{ult}	μ	C_A	C_B	λ	eq1	eq2	SOLVE FOR = 0, λ	nomogram	L_{cr}	i	λ_e	λ	λ_{rel}	M_{Rk}	
N	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	mm	mm	mm	mm	[-]	Nmm	
O1	1447584	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	127.53	76.41	91.27	1.19	423174646
O2	1339919	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	127.53	76.41	97.40	1.27	423174646
O3	1244369	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	127.53	76.41	103.29	1.35	423174646
A1	172002	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	49.04	76.41	142.41	1.86	40386575
A2	155573	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	49.04	76.41	151.98	1.99	40386575
A3	141740	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	49.04	76.41	161.17	2.11	40386575
A4	409493	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	65.79	76.41	106.15	1.39	82663170
A5	375537	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	65.79	76.41	113.28	1.48	82663170
A6	346048	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	65.79	76.41	120.13	1.57	82663170
A7	1032536	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	91.77	76.41	76.10	1.00	192918360
A8	968521	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	91.77	76.41	81.21	1.06	192918360
A9	909947	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	91.77	76.41	86.13	1.13	192918360
A10	1937028	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	118.66	76.41	58.85	0.77	341841936
A11	1852086	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	118.66	76.41	62.81	0.82	341841936
A12	1771294	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	118.66	76.41	66.61	0.87	341841936
O1-NL	1649141	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	127.53	76.41	91.27	1.19	423174646
O2-NL	1519941	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	127.53	76.41	97.40	1.27	423174646
O3-NL	1405168	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	127.53	76.41	103.29	1.35	423174646
A1-NL	190076	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	49.04	76.41	142.41	1.86	40386575
A2-NL	170842	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	49.04	76.41	151.98	1.99	40386575
A3-NL	154800	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	49.04	76.41	161.17	2.11	40386575
A4-NL	462835	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	65.79	76.41	106.15	1.39	82663170
A5-NL	421980	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	65.79	76.41	113.28	1.48	82663170
A6-NL	386702	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	65.79	76.41	120.13	1.57	82663170
A7-NL	1180162	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	91.77	76.41	76.10	1.00	192918360
A8-NL	1105774	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	91.77	76.41	81.21	1.06	192918360
A9-NL	1036770	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	91.77	76.41	86.13	1.13	192918360
A10-NL	2190255	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	118.66	76.41	58.85	0.77	341841936
A11-NL	2101455	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	118.66	76.41	62.81	0.82	341841936
A12-NL	2014564	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	118.66	76.41	66.61	0.87	341841936
O1-V	746778	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	127.53	76.41	91.27	1.19	423174646
O2-V	699288	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	127.53	76.41	97.40	1.27	423174646
O3-V	657544	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	127.53	76.41	103.29	1.35	423174646
A1-V	96373	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	49.04	76.41	142.41	1.86	40386575
A2-V	89173	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	49.04	76.41	151.98	1.99	40386575
A3-V	82952	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	49.04	76.41	161.17	2.11	40386575
A4-V	214920	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	65.79	76.41	106.15	1.39	82663170
A5-V	200468	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	65.79	76.41	113.28	1.48	82663170
A6-V	187834	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	65.79	76.41	120.13	1.57	82663170
A7-V	529866	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	91.77	76.41	76.10	1.00	192918360
A8-V	498794	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	91.77	76.41	81.21	1.06	192918360
A9-V	471276	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	91.77	76.41	86.13	1.13	192918360
A10-V	1022414	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	118.66	76.41	58.85	0.77	341841936
A11-V	970046	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	118.66	76.41	62.81	0.82	341841936
A12-V	923161	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	118.66	76.41	66.61	0.87	341841936
O1-H	1328401	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	127.53	76.41	91.27	1.19	423174646
O2-H	1177007	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	127.53	76.41	97.40	1.27	423174646
O3-H	1045783	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	127.53	76.41	103.29	1.35	423174646
A1-H	142718	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	49.04	76.41	142.41	1.86	40386575
A2-H	116654	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	49.04	76.41	151.98	1.99	40386575
A3-H	95285	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	49.04	76.41	161.17	2.11	40386575
A4-H	373130	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	65.79	76.41	106.15	1.39	82663170
A5-H	326399	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	65.79	76.41	113.28	1.48	82663170
A6-H	286672	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	65.79	76.41	120.13	1.57	82663170
A7-H	991244	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	91.77	76.41	76.10	1.00	192918360
A8-H	910711	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	91.77	76.41	81.21	1.06	192918360
A9-H	838053	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	91.77	76.41	86.13	1.13	192918360
A10-H	1893858	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	118.66	76.41	58.85	0.77	341841936
A11-H	1789709	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	118.66	76.41	62.81	0.82	341841936
A12-H	1691406	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	118.66	76.41	66.61	0.87	341841936
O1-VH	694424	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	127.53	76.41	91.27	1.19	423174646
O2-VH	625908	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	127.53	76.41	97.40	1.27	423174646
O3-VH	565757	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	127.53	76.41	103.29	1.35	423174646
A1-VH	81317	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	49.04	76.41	142.41	1.86	40386575
A2-VH	68367	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	49.04	76.41	151.98	1.99	40386575
A3-VH	57255	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	49.04	76.41	161.17	2.11	40386575
A4-VH	198327	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	65.79	76.41	106.15	1.39	82663170
A5-VH	177311	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	65.79	76.41	113.28	1.48	82663170
A6-VH	158														

Modified EC1993-1-1- with Nomogram															
variable for solve function		$W_i \cdot f_{yd}$	$A^* f_{yd}$			$(E_i/L_i) / (\mu(E_i/L_i))$		$C_A \cdot C_B \cdot \lambda^2 \sin(\lambda)$	$(C_A + C_B) \cdot \lambda \cos(\lambda) + \sin(\lambda)$	eq1-eq2=0	$\pi/\lambda (K_0)$	$\pi/\lambda^3 h$	$\pi^2 E_i / L_{cr}^2$	$\lambda' = \sqrt{(F_{Ri}/F_{cr})}$	
F_{ult}	α	M_{Rk}	N_{Rk}	μ	C_A	C_B	λ	eq1	eq2	SOLVE FOR = 0, λ	nomogram	L_{cr}	N_{cr}	λ'	
N	[-]	Nmm	N	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	mm	N	[-]	
O1	1447584	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	2644346	1.19
O2	1339919	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	2321877	1.27
O3	1244369	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	2064550	1.35
A1	172002	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	246298	1.86
A2	155573	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	216263	1.99
A3	141740	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	192295	2.11
A4	409493	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	677645	1.39
A5	375537	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	595008	1.48
A6	346048	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	529065	1.57
A7	1032536	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	2203124	1.00
A8	968521	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	1934460	1.06
A9	909947	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	1720069	1.13
A10	1937028	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	5524428	0.77
A11	1852086	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	4850741	0.82
A12	1771294	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	4313148	0.87
O1-NL	1649141	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	2644346	1.19
O2-NL	1519941	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	2321877	1.27
O3-NL	1405168	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	2064550	1.35
A1-NL	190076	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	246298	1.86
A2-NL	170842	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	216263	1.99
A3-NL	154800	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	192295	2.11
A4-NL	462835	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	677645	1.39
A5-NL	421980	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	595008	1.48
A6-NL	386702	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	529065	1.57
A7-NL	1180162	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	2203124	1.00
A8-NL	1105774	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	1934460	1.06
A9-NL	1036770	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	1720069	1.13
A10-NL	2190255	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	5524428	0.77
A11-NL	2101455	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	4850741	0.82
A12-NL	2014564	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	4313148	0.87
O1-V	746778	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	2644346	1.19
O2-V	699288	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	2321877	1.27
O3-V	657544	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	2064550	1.35
A1-V	96373	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	246298	1.86
A2-V	89173	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	216263	1.99
A3-V	82952	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	192295	2.11
A4-V	214920	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	677645	1.39
A5-V	200468	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	595008	1.48
A6-V	187834	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	529065	1.57
A7-V	529866	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	2203124	1.00
A8-V	498794	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	1934460	1.06
A9-V	471276	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	1720069	1.13
A10-V	1022414	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	5524428	0.77
A11-V	970046	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	4850741	0.82
A12-V	923161	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	4313148	0.87
O1-H	1328401	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	2644346	1.19
O2-H	1177007	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	2321877	1.27
O3-H	1045783	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	2064550	1.35
A1-H	142718	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	246298	1.86
A2-H	116654	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	216263	1.99
A3-H	95285	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	192295	2.11
A4-H	373130	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	677645	1.39
A5-H	326399	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	595008	1.48
A6-H	286672	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	529065	1.57
A7-H	991244	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	2203124	1.00
A8-H	910711	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	1934460	1.06
A9-H	838053	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	1720069	1.13
A10-H	1893858	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	5524428	0.77
A11-H	1789709	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	4850741	0.82
A12-H	1691406	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	4313148	0.87
O1-VH	694424	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	11639	2644346	1.19
O2-VH	625908	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	12421	2321877	1.27
O3-VH	565757	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	13173	2064550	1.35
A1-VH	81317	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	246298	1.86
A2-VH	68367	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00	2.48	7453	216263	1.99
A3-VH	57255	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00	2.63	7904	192295	2.11
A4-VH	198327	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00	2.33	6984	677645	1.39
A5-VH	177311	0.34	82663170	1307820	6	1E+104	0.250	1.26</							

Modified EC1993-1-1 with Nomogram													
$\Phi = 0.5(1 + \alpha \cdot (\lambda - 0.2) + \lambda^2)$		$\chi = 1 / (\Phi + \sqrt{(\Phi^2 - \lambda^2)})$	M_{Rd} / Y_{M0}	N_{Rd} / Y_{M0}	$1 / (1 - ((F_{ult} / F_{Rd}) \cdot \chi \cdot \lambda^2))$			factor * N_{Ed}			$F_{ult} \cdot h^3 / 2^2 \cdot C_m$	$F / N_{Rd} + (n/n-1) \cdot F_{tot} \cdot e^* / M_{Rd}$	
Φ	χ		Y_{M0}	M_{Rd}	N_{Rd}	k	N_{Ed}	factor	F_{lean}	ϕF	c_m	M_{Ed}	SOLVE FOR = 1, F_{ult}
[-]	[-]	[-]		Nmm	N	[-]	N	[-]	N	N		N	[-]
O1	1.38	0.48	1.0	423174646	3772585	1.36	1447584	0.5	723792	5285.8	0.9	39643722	1.00
O2	1.50	0.44	1.0	423174646	3772585	1.34	1339919	0.5	669960	4892.7	0.9	36695195	1.00
O3	1.61	0.40	1.0	423174646	3772585	1.32	1244369	0.5	622184	4543.8	0.9	34078448	1.00
A1	2.52	0.24	1.0	40386575	855550	1.20	172002	0.5	86001	702.2	0.9	3159881	1.00
A2	2.78	0.21	1.0	40386575	855550	1.18	155573	0.5	77787	635.1	0.9	2858066	1.00
A3	3.05	0.19	1.0	40386575	855550	1.16	141740	0.5	70870	578.7	0.9	2603928	1.00
A4	1.67	0.39	1.0	82663170	1307820	1.30	409493	0.5	204747	1671.7	0.9	7522872	1.00
A5	1.82	0.35	1.0	82663170	1307820	1.28	375537	0.5	187768	1533.1	0.9	6899055	1.00
A6	1.97	0.32	1.0	82663170	1307820	1.26	346048	0.5	173024	1412.7	0.9	6357311	1.00
A7	1.13	0.60	1.0	192918360	2185380	1.39	1032536	0.5	516268	4215.3	0.9	18968898	1.00
A8	1.21	0.56	1.0	192918360	2185380	1.39	968521	0.5	484261	3954.0	0.9	17792870	1.00
A9	1.29	0.52	1.0	192918360	2185380	1.38	909947	0.5	454973	3714.8	0.9	16716788	1.00
A10	0.89	0.74	1.0	341841936	3277360	1.35	1937028	0.5	968514	7907.9	0.9	35585486	1.00
A11	0.94	0.71	1.0	341841936	3277360	1.37	1852086	0.5	926043	7561.1	0.9	34024995	1.00
A12	0.99	0.68	1.0	341841936	3277360	1.39	1771294	0.5	885647	7231.3	0.9	32540753	1.00
O1-NL	1.38	0.48	1.0	423174646	3772585	1.43	1649141	0.0	0	6021.8	0.9	30109052	1.00
O2-NL	1.50	0.44	1.0	423174646	3772585	1.40	1519941	0.0	0	5550.0	0.9	27750199	1.00
O3-NL	1.61	0.40	1.0	423174646	3772585	1.38	1405168	0.0	0	5130.9	0.9	25654742	1.00
A1-NL	2.52	0.24	1.0	40386575	855550	1.22	190076	0.0	0	776.0	0.9	2327948	1.00
A2-NL	2.78	0.21	1.0	40386575	855550	1.20	170842	0.0	0	697.5	0.9	2092375	1.00
A3-NL	3.05	0.19	1.0	40386575	855550	1.18	154800	0.0	0	632.0	0.9	1895906	1.00
A4-NL	1.67	0.39	1.0	82663170	1307820	1.36	462835	0.0	0	1889.5	0.9	5668549	1.00
A5-NL	1.82	0.35	1.0	82663170	1307820	1.33	421980	0.0	0	1722.7	0.9	5168175	1.00
A6-NL	1.97	0.32	1.0	82663170	1307820	1.30	386702	0.0	0	1578.7	0.9	4736115	1.00
A7-NL	1.13	0.60	1.0	192918360	2185380	1.47	1180162	0.0	0	4818.0	0.9	14453974	1.00
A8-NL	1.21	0.56	1.0	192918360	2185380	1.47	1105774	0.0	0	4514.3	0.9	13542905	1.00
A9-NL	1.29	0.52	1.0	192918360	2185380	1.46	1036770	0.0	0	4232.6	0.9	12697786	1.00
A10-NL	0.89	0.74	1.0	341841936	3277360	1.42	2190255	0.0	0	8941.7	0.9	26825040	1.00
A11-NL	0.94	0.71	1.0	341841936	3277360	1.44	2101455	0.0	0	8579.2	0.9	25737457	1.00
A12-NL	0.99	0.68	1.0	341841936	3277360	1.46	2014564	0.0	0	8224.4	0.9	24673269	1.00
O1-V	1.38	0.48	1.0	423174646	3772585	1.16	746778	5.0	3733889	2726.8	0.9	81805417	1.00
O2-V	1.50	0.44	1.0	423174646	3772585	1.15	699288	5.0	3496441	2553.4	0.9	76603191	1.00
O3-V	1.61	0.40	1.0	423174646	3772585	1.15	657544	5.0	3287721	2401.0	0.9	72030359	1.00
A1-V	2.52	0.24	1.0	40386575	855550	1.10	96373	5.0	481866	393.4	0.9	7081955	1.00
A2-V	2.78	0.21	1.0	40386575	855550	1.10	89173	5.0	445864	364.0	0.9	6552837	1.00
A3-V	3.05	0.19	1.0	40386575	855550	1.09	82952	5.0	414761	338.7	0.9	6095713	1.00
A4-V	1.67	0.39	1.0	82663170	1307820	1.14	214920	5.0	1074598	877.4	0.9	15793300	1.00
A5-V	1.82	0.35	1.0	82663170	1307820	1.13	200468	5.0	1002339	818.4	0.9	14731314	1.00
A6-V	1.97	0.32	1.0	82663170	1307820	1.13	187834	5.0	939170	766.8	0.9	13802928	1.00
A7-V	1.13	0.60	1.0	192918360	2185380	1.17	529866	5.0	2649331	2163.2	0.9	38937061	1.00
A8-V	1.21	0.56	1.0	192918360	2185380	1.17	498794	5.0	2493970	2036.3	0.9	36653724	1.00
A9-V	1.29	0.52	1.0	192918360	2185380	1.17	471276	5.0	2356380	1924.0	0.9	34631566	1.00
A10-V	0.89	0.74	1.0	341841936	3277360	1.16	1022414	5.0	5112072	4174.0	0.9	75131803	1.00
A11-V	0.94	0.71	1.0	341841936	3277360	1.17	970046	5.0	4850232	3960.2	0.9	71283556	1.00
A12-V	0.99	0.68	1.0	341841936	3277360	1.17	923161	5.0	4615805	3768.8	0.9	67838203	1.00
O1-H	1.38	0.48	1.0	423174646	3772585	1.32	1328401	0.5	664201	4850.6	0.9	67629765	1.00
O2-H	1.50	0.44	1.0	423174646	3772585	1.29	1177007	0.5	588504	4297.8	0.9	79108673	1.00
O3-H	1.61	0.40	1.0	423174646	3772585	1.26	1045783	0.5	522891	3818.7	0.9	91139940	1.00
A1-H	2.52	0.24	1.0	40386575	855550	1.16	142718	0.5	71359	582.6	0.9	9371895	1.00
A2-H	2.78	0.21	1.0	40386575	855550	1.13	116654	0.5	58327	476.2	0.9	12280663	1.00
A3-H	3.05	0.19	1.0	40386575	855550	1.10	95285	0.5	47642	389.0	0.9	15250495	1.00
A4-H	1.67	0.39	1.0	82663170	1307820	1.27	373130	0.5	186565	1523.3	0.9	13604834	1.00
A5-H	1.82	0.35	1.0	82663170	1307820	1.24	326399	0.5	163199	1332.5	0.9	16121332	1.00
A6-H	1.97	0.32	1.0	82663170	1307820	1.21	286672	0.5	143336	1170.3	0.9	18766510	1.00
A7-H	1.13	0.60	1.0	192918360	2185380	1.37	991244	0.5	495622	4046.7	0.9	24960307	1.00
A8-H	1.21	0.56	1.0	192918360	2185380	1.36	910711	0.5	455356	3718.0	0.9	26855838	1.00
A9-H	1.29	0.52	1.0	192918360	2185380	1.34	838053	0.5	419027	3421.3	0.9	28896023	1.00
A10-H	0.89	0.74	1.0	341841936	3277360	1.34	1893858	0.5	946929	7731.6	0.9	41542389	1.00
A11-H	0.94	0.71	1.0	341841936	3277360	1.36	1789709	0.5	894855	7306.5	0.9	43004058	1.00
A12-H	0.99	0.68	1.0	341841936	3277360	1.36	1691406	0.5	845703	6905.1	0.9	44573108	1.00
O1-VH	1.38	0.48	1.0	423174646	3772585	1.14	694424	5.0	3472120	2535.7	0.9	107320348	1.00
O2-VH	1.50	0.44	1.0	423174646	3772585	1.13	625908	5.0	3129538	2285.5	0.9	115439736	1.00
O3-VH	1.61	0.40	1.0	423174646	3772585	1.12	565757	5.0	2828784	2065.9	0.9	124475561	1.00
A1-VH	2.52	0.24	1.0	40386575	855550	1.08	81317	5.0	406583	332.0	0.9	12725530	1.00
A2-VH	2.78	0.21	1.0	40386575	855550	1.07	68367	5.0	341835	279.1	0.9	15148931	1.00
A3-VH	3.05	0.19	1.0	40386575	855550	1.06	57255	5.0	286274	233.7	0.9	17707349	1.00
A4-VH	1.67	0.39	1.0	82663170	1307820	1.13	198327	5.0	991634	809.7	0.9	21323990	1.00
A5-VH	1.82	0.35	1.0	82663170	1307820	1.12	177311	5.0	886556	723.9	0.9	23154657	1.00
A6-VH	1.97	0.32	1.0	82663170	1307820	1.11	158981	5.0	794903	649.0	0.9	25182638	1.00
A7-VH	1.13	0.60	1.0	192918360	2185380	1.16	512184	5.0	2560921	2091.0	0.9	44387691	1.00
A8-VH	1.21	0.56	1.0	192918360	2185380	1.16	473838	5.0	2369191	1934.4	0.9	44944860	1.00
A9-VH	1.29	0.52	1.0	192918360	2185380	1.15	439863	5.0	2199313	1795.7	0.9	45823169	1.00
A10-VH	0.89	0.74	1.0	341841936	3277360	1.16	1002154	5.0	5015768	4095.4	0.9	80466431	1.00
A11-VH	0.94	0.71	1.0	341841936	3277360	1.16	942610	5.0	4713048	3848.2	0.9	79392383	1.00
A12-VH	0.99	0.68	1.0	341841936	3277360	1.16	888329	5.0	4441643	3626.6	0.9	78778551	1.00
O1-NLH	1.38	0.48	1.0	423174646	3772585	1.38	1503290	0.0	0	5489.2	0.9	58696203	1.00
O2-NLH	1.50	0.44	1.0	423174646	3772585	1.33	1323252	0.0	0	4831.8	0.9	71034163	1.00
O3-NLH	1.61	0.40	1.0	423174646	3772585	1.30	1168495	0.0	0	4266.7	0.9	83833697	1.00
A1-NLH	2.52	0.24	1.0	40386575	855550	1.18	156775	0.0	0	640.0	0.9	8670099	1.00
A2-NLH	2.78	0.21	1.0	40386575	855550	1.14	127166	0.0	0	519.2	0.9	11682459	1.00
A3-NLH	3.05	0.19	1.0	40386575	855550	1.11	103220	0.0	0	421.4	0.9	14764182	1.00
A4-NLH	1.67	0.39	1.0	82663170	1307820	1.31	419285	0.0	0	1711.7	0.9	11885167	1.00
A													

Current EC1993-1-1 with LBA															
variable for solve function		W^*f_{jd}	A^*f_{jd}	LBA	$\lambda = \sqrt{(N_{Rk}/N_{cr})}$	$\Phi = 0.5(1 + \alpha(\lambda - 0.2) + \lambda^2)$	$\chi = 1 / (\Phi + \sqrt{\Phi^2 - \lambda^2})$		M_{Rk}/Y_{M0}	N_{Rk}/Y_{M0}	$1 / (1 - (F_{ult}/N_{Rk}) * \chi * \lambda^2)$				
N_{Rk}	α	M_{Rk}	N_{Rk}	N_{cr}	λ	Φ	χ	Y_{M0}	M_{Rd}	N_{Rd}	k	N_{Ed}	ϕF	c_m	
N	[-]	Nmm	N	N	[-]	[-]	X		[-]	Nmm	N	[-]	N	[-]	
01	1230615	0.34	423174646	3772585	1789800	1.45	1.77	0.36	1.0	423174646	3772585	1.33	1230615	4493.6	0.9
02	1123588	0.34	423174646	3772585	1576800	1.55	1.93	0.33	1.0	423174646	3772585	1.30	1123588	4102.8	0.9
03	1028403	0.34	423174646	3772585	1401000	1.64	2.09	0.30	1.0	423174646	3772585	1.28	1028403	3755.2	0.9
A1	135630	0.34	40386575	855550	168970	2.25	3.38	0.17	1.0	40386575	855550	1.16	135630	553.7	0.9
A2	121045	0.34	40386575	855550	147850	2.41	3.77	0.15	1.0	40386575	855550	1.14	121045	494.2	0.9
A3	108978	0.34	40386575	855550	130990	2.56	4.17	0.13	1.0	40386575	855550	1.13	108978	444.9	0.9
A4	338450	0.34	82663170	1307820	461940	1.68	2.17	0.28	1.0	82663170	1307820	1.26	338450	1381.7	0.9
A5	305986	0.34	82663170	1307820	405600	1.80	2.38	0.25	1.0	82663170	1307820	1.24	305986	1249.2	0.9
A6	278058	0.34	82663170	1307820	359850	1.91	2.61	0.23	1.0	82663170	1307820	1.21	278058	1135.2	0.9
A7	910477	0.34	192918360	2185380	1471200	1.22	1.42	0.47	1.0	192918360	2185380	1.41	910477	3717.0	0.9
A8	843895	0.34	192918360	2185380	1302500	1.30	1.53	0.43	1.0	192918360	2185380	1.39	843895	3445.2	0.9
A9	781612	0.34	192918360	2185380	1160100	1.37	1.64	0.39	1.0	192918360	2185380	1.36	781612	3190.9	0.9
A10	1789986	0.34	341841936	3277360	3582200	0.96	1.09	0.62	1.0	341841936	3277360	1.45	1789986	7307.6	0.9
A11	1702045	0.34	341841936	3277360	3206500	1.01	1.15	0.59	1.0	341841936	3277360	1.46	1702045	6948.5	0.9
A12	1611243	0.34	341841936	3277360	2871200	1.07	1.22	0.55	1.0	341841936	3277360	1.45	1611243	6577.9	0.9
01-NL	1632663	0.34	423174646	3772585	2601000	1.20	1.40	0.48	1.0	423174646	3772585	1.43	1632663	5961.6	0.9
02-NL	1514411	0.34	423174646	3772585	2308900	1.28	1.50	0.44	1.0	423174646	3772585	1.40	1514411	5529.8	0.9
03-NL	1404397	0.34	423174646	3772585	2062900	1.35	1.61	0.40	1.0	423174646	3772585	1.38	1404397	5128.1	0.9
A1-NL	189527	0.34	40386575	855550	245420	1.87	2.53	0.24	1.0	40386575	855550	1.22	189527	773.7	0.9
A2-NL	170978	0.34	40386575	855550	216470	1.99	2.78	0.21	1.0	40386575	855550	1.20	170978	698.0	0.9
A3-NL	155179	0.34	40386575	855550	192850	2.11	3.04	0.19	1.0	40386575	855550	1.18	155179	633.5	0.9
A4-NL	459740	0.34	82663170	1307820	671120	1.40	1.68	0.38	1.0	82663170	1307820	1.36	459740	1876.9	0.9
A5-NL	421397	0.34	82663170	1307820	593880	1.48	1.82	0.35	1.0	82663170	1307820	1.33	421397	1720.3	0.9
A6-NL	387120	0.34	82663170	1307820	529820	1.57	1.97	0.32	1.0	82663170	1307820	1.30	387120	1580.4	0.9
A7-NL	1163564	0.34	192918360	2185380	2139200	1.01	1.15	0.59	1.0	192918360	2185380	1.47	1163564	4750.2	0.9
A8-NL	1097722	0.34	192918360	2185380	1907900	1.07	1.22	0.55	1.0	192918360	2185380	1.47	1097722	4481.4	0.9
A9-NL	1032774	0.34	192918360	2185380	1708500	1.13	1.30	0.52	1.0	192918360	2185380	1.45	1032774	4216.3	0.9
A10-NL	2151840	0.34	341841936	3277360	5214800	0.79	0.92	0.73	1.0	341841936	3277360	1.43	2151840	8784.9	0.9
A11-NL	2078624	0.34	341841936	3277360	4699400	0.84	0.96	0.70	1.0	341841936	3277360	1.45	2078624	8485.9	0.9
A12-NL	1999644	0.34	341841936	3277360	4230400	0.88	1.00	0.67	1.0	341841936	3277360	1.47	1999644	8163.5	0.9
01-V	375672	0.34	423174646	3772585	467540	2.84	4.98	0.11	1.0	423174646	3772585	1.10	375672	1371.8	0.9
02-V	334114	0.34	423174646	3772585	407140	3.04	5.62	0.10	1.0	423174646	3772585	1.09	334114	1220.0	0.9
03-V	299573	0.34	423174646	3772585	358790	3.24	6.27	0.09	1.0	423174646	3772585	1.08	299573	1093.9	0.9
A1-V	38348	0.34	40386575	855550	44217.4	4.40	10.89	0.05	1.0	40386575	855550	1.04	38348	156.6	0.9
A2-V	33631	0.34	40386575	855550	38232.7	4.73	12.46	0.04	1.0	40386575	855550	1.04	33631	137.3	0.9
A3-V	29890	0.34	40386575	855550	33597.7	5.05	14.06	0.04	1.0	40386575	855550	1.03	29890	122.0	0.9
A4-V	99844	0.34	82663170	1307820	120780	3.29	6.44	0.08	1.0	82663170	1307820	1.07	99844	407.6	0.9
A5-V	88269	0.34	82663170	1307820	104800	3.53	7.31	0.07	1.0	82663170	1307820	1.07	88269	360.4	0.9
A6-V	78855	0.34	82663170	1307820	92220.2	3.77	8.20	0.06	1.0	82663170	1307820	1.06	78855	321.9	0.9
A7-V	294507	0.34	192918360	2185380	383710	2.39	3.72	0.15	1.0	192918360	2185380	1.13	294507	1202.3	0.9
A8-V	264333	0.34	192918360	2185380	335900	2.55	4.15	0.13	1.0	192918360	2185380	1.12	264333	1079.1	0.9
A9-V	238436	0.34	192918360	2185380	296720	2.71	4.61	0.12	1.0	192918360	2185380	1.11	238436	973.4	0.9
A10-V	652575	0.34	341841936	3277360	931330	1.88	2.54	0.23	1.0	341841936	3277360	1.20	652575	2664.1	0.9
A11-V	596523	0.34	341841936	3277360	824800	1.99	2.79	0.21	1.0	341841936	3277360	1.18	596523	2435.3	0.9
A12-V	544789	0.34	341841936	3277360	732480	2.12	3.06	0.19	1.0	341841936	3277360	1.16	544789	2224.1	0.9
01-H	1127668	0.34	423174646	3772585	1789800	1.45	1.77	0.36	1.0	423174646	3772585	1.29	1127668	4117.7	0.9
02-H	985941	0.34	423174646	3772585	1576800	1.55	1.93	0.33	1.0	423174646	3772585	1.26	985941	3600.1	0.9
03-H	864076	0.34	423174646	3772585	1401000	1.64	2.09	0.30	1.0	423174646	3772585	1.22	864076	3155.2	0.9
A1-H	112814	0.34	40386575	855550	168970	2.25	3.38	0.17	1.0	40386575	855550	1.13	112814	460.6	0.9
A2-H	91124	0.34	40386575	855550	147850	2.41	3.77	0.15	1.0	40386575	855550	1.10	91124	372.0	0.9
A3-H	73663	0.34	40386575	855550	130990	2.56	4.17	0.13	1.0	40386575	855550	1.08	73663	300.7	0.9
A4-H	308382	0.34	82663170	1307820	461940	1.68	2.17	0.28	1.0	82663170	1307820	1.23	308382	1259.9	0.9
A5-H	266138	0.34	82663170	1307820	405600	1.80	2.38	0.25	1.0	82663170	1307820	1.20	266138	1086.5	0.9
A6-H	230738	0.34	82663170	1307820	359850	1.91	2.61	0.23	1.0	82663170	1307820	1.17	230738	942.0	0.9
A7-H	872399	0.34	192918360	2185380	1471200	1.22	1.42	0.47	1.0	192918360	2185380	1.38	872399	3561.6	0.9
A8-H	791806	0.34	192918360	2185380	1302500	1.30	1.53	0.43	1.0	192918360	2185380	1.35	791806	3232.5	0.9
A9-H	718382	0.34	192918360	2185380	1160100	1.37	1.64	0.39	1.0	192918360	2185380	1.32	718382	2932.8	0.9
A10-H	1746629	0.34	341841936	3277360	3582200	0.96	1.09	0.62	1.0	341841936	3277360	1.44	1746629	7130.6	0.9
A11-H	1640260	0.34	341841936	3277360	3206500	1.01	1.15	0.59	1.0	341841936	3277360	1.43	1640260	6696.3	0.9
A12-H	1533582	0.34	341841936	3277360	2871200	1.07	1.22	0.55	1.0	341841936	3277360	1.42	1533582	6260.8	0.9
01-VH	348701	0.34	423174646	3772585	467540	2.84	4.98	0.11	1.0	423174646	3772585	1.09	348701	1273.3	0.9
02-VH	298492	0.34	423174646	3772585	407140	3.04	5.62	0.10	1.0	423174646	3772585	1.08	298492	1089.9	0.9
03-VH	257347	0.34	423174646	3772585	358790	3.24	6.27	0.09	1.0	423174646	3772585	1.07	257347	939.7	0.9
A1-VH	32384	0.34	40386575	855550	44217.4	4.40	10.89	0.05	1.0	40386575	855550	1.04	32384	132.2	0.9
A2-VH	25836	0.34	40386575	855550	38232.7	4.73	12.46	0.04	1.0	40386575	855550	1.03	25836	105.5	0.9
A3-VH	20700	0.34	40386575	855550	33597.7	5.05	14.06	0.04	1.0	40386575	855550	1.02	20700	84.5	0.9
A4-VH	92053	0.34	82663170	1307820	120780	3.29	6.44	0.08	1.0	82663170	1307820	1.07	92053	375.8	0.9
A5-VH	78023	0.34	82663170	1307820	104800	3.53	7.31	0.07	1.0	82663170	1307820	1.06	78023	318.5	0.9
A6-VH	66736	0.34	82663170	1307820	92220.2	3.77	8.20	0.06	1.0	82663170	1307820	1.05	66736	272.4	0.9
A7-VH	284204	0.34	192918360												

	$(2 \cdot \varphi N_{Ed} + (Q/F) \cdot \varphi N_{Ed} + H) \cdot h \cdot C$	$N_{Ed}/N_{Rd} + (n/n - 1) \cdot F_{t,Ed} \cdot e^* / M_{Rd}$
M_{Ed}	SOLVE FOR = 1, F_{ult}	
N_{mm}	[-]	

O1-NL	29808215	1.00
O2-NL	27649228	1.00
O3-NL	25640657	1.00
A1-NL	2321225	1.00
A2-NL	2094039	1.00
A3-NL	1900542	1.00
A4-NL	5630636	1.00
A5-NL	5161033	1.00
A6-NL	4741232	1.00
A7-NL	14250686	1.00
A8-NL	13444295	1.00
A9-NL	12648842	1.00
A10-NL	26354556	1.00
A11-NL	25457840	1.00
A12-NL	24490539	1.00

O1-V	41152797	1.00
O2-V	36600322	1.00
O3-V	32816555	1.00
A1-V	2818005	1.00
A2-V	2471377	1.00
A3-V	2196429	1.00
A4-V	7337019	1.00
A5-V	6486391	1.00
A6-V	5794627	1.00
A7-V	21641745	1.00
A8-V	19424415	1.00
A9-V	17521429	1.00
A10-V	47954266	1.00
A11-V	43835315	1.00
A12-V	40033644	1.00

O1-H	62132464	1.00
O2-H	73876098	1.00
O3-H	86163689	1.00
A1-H	8822519	1.00
A2-H	11799057	1.00
A3-H	14853282	1.00
A4-H	12415344	1.00
A5-H	15014262	1.00
A6-H	17738920	1.00
A7-H	22776986	1.00
A8-H	24671400	1.00
A9-H	26697518	1.00
A10-H	38837628	1.00
A11-H	40258498	1.00
A12-H	41673699	1.00

O1-VH	69448275	1.00
O2-VH	79573164	1.00
O3-VH	90690997	1.00
A1-VH	9129730	1.00
A2-VH	12023539	1.00
A3-VH	15021114	1.00
A4-VH	13514502	1.00
A5-VH	15858523	1.00
A6-VH	18404063	1.00
A7-VH	27634679	1.00
A8-VH	28540891	1.00
A9-VH	29819803	1.00
A10-VH	53717169	1.00
A11-VH	52620859	1.00
A12-VH	51919186	1.00

O1-NLH	58424967	1.00
O2-NLH	70947978	1.00
O3-NLH	83822311	1.00
A1-NLH	8664696	1.00
A2-NLH	11683655	1.00
A3-NLH	14767143	1.00
A4-NLH	11851361	1.00
A5-NLH	14577315	1.00
A6-NLH	17393839	1.00
A7-NLH	20380012	1.00
A8-NLH	22699023	1.00
A9-NLH	25070405	1.00
A10-NLH	32467869	1.00
A11-NLH	34652845	1.00
A12-NLH	36791098	1.00

Current EC1993-1-1 with Nomogram												
variable for solve function												
N_{ult}	α	M_{Rk}	F_{Rk}	μ	C_A	C_B	λ	eq1	eq2	eq1-eq2=0	SOLVE FOR = 0	nomogram
[-]	Nmm	N	N									
1580631	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1462127	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1356185	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
184341	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
166262	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
151080	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
444948	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
407273	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
374491	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1127622	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1059474	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
996091	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
2091391	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
2008789	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1928205	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1649141	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1519941	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1405168	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
190076	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
170842	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
154800	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
462835	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
421980	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
386702	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1180162	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1105774	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1036770	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
2190255	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
2101455	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
2014564	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1179849	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1112924	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1051253	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
146344	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
134913	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
124901	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
336746	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
315071	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
295459	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
829250	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
790807	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
754548	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1528214	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1478728	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1430805	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1444403	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1276880	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1131676	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
152348	0.34	40386575	855550	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
124042	0.34	40386575	855550	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
100983	0.34	40386575	855550	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
403930	0.34	82663170	1307820	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
352230	0.34	82663170	1307820	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
308418	0.34	82663170	1307820	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1080089	0.34	192918360	2185380	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
992853	0.34	192918360	2185380	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
913327	0.34	192918360	2185380	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
2042745	0.34	341841936	3277360	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
1937877	0.34	341841936	3277360	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
1836760	0.34	341841936	3277360	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
1089831	0.34	423174646	3772585	6	1E+104	0.167	1.35	2.9615E+103	2.9615E+103	0.00		2.33
986082	0.34	423174646	3772585	6	1E+104	0.250	1.26	3.812E+103	3.812E+103	0.00		2.48
892381	0.34	423174646	3772585	6	1E+104	0.333	1.19	4.4047E+103	4.4047E+103	0.00		2.63
122287	0.34	40386575	855550	6								

Current EC1993-1-1 with Yura													
variable for solve function		W^*f_{jd}	A^*f_{jd}	$K_0 \cdot \sqrt{(\sum F + \sum Q) / \sum F}$	K^*h	$\pi^2 EI / L_{cr}^2$	$\lambda' = \sqrt{F_{Rk} / F_{cr}}$	$\Phi = 0.5(1 + \alpha \cdot (\lambda - 0.2) + \lambda^2)$	$\chi = 1 / (\Phi + \sqrt{\Phi^2 - \lambda^2})$		M_{Rk} / γ_{M0}	N_{Rk} / γ_{M0}	
F_{ult}	α	M_{Rk}	N_{Rk}	K	L_{cr}	N_{cr}	λ'	Φ	χ	γ_{M0}	M_{Rd}	N_{Rd}	
N	[-]	Nmm	N	[-]	mm	N	[-]	[-]	[-]		Nmm	N	
O1	1217580	0.34	423174646	3772585	2.85	14255	1762898	1.46	1.78	0.36	1.0	423174646	3772585
O2	1108386	0.34	423174646	3772585	3.04	15213	1547918	1.56	1.95	0.32	1.0	423174646	3772585
O3	1014549	0.34	423174646	3772585	3.23	16133	1376367	1.66	2.12	0.29	1.0	423174646	3772585
A1	132386	0.34	40386575	855550	2.85	8553	164199	2.28	3.46	0.17	1.0	40386575	855550
A2	118447	0.34	40386575	855550	3.04	9128	144175	2.44	3.85	0.15	1.0	40386575	855550
A3	106942	0.34	40386575	855550	3.23	9680	128197	2.58	4.24	0.13	1.0	40386575	855550
A4	332739	0.34	82663170	1307820	2.85	8553	451763	1.70	2.20	0.28	1.0	82663170	1307820
A5	300648	0.34	82663170	1307820	3.04	9128	396672	1.82	2.42	0.25	1.0	82663170	1307820
A6	273569	0.34	82663170	1307820	3.23	9680	352710	1.93	2.65	0.22	1.0	82663170	1307820
A7	909562	0.34	192918360	2185380	2.85	8553	1468749	1.22	1.42	0.47	1.0	192918360	2185380
A8	838508	0.34	192918360	2185380	3.04	9128	1289640	1.30	1.53	0.43	1.0	192918360	2185380
A9	775450	0.34	192918360	2185380	3.23	9680	1146713	1.38	1.65	0.39	1.0	192918360	2185380
A10	1811409	0.34	341841936	3277360	2.85	8553	3682952	0.94	1.07	0.63	1.0	341841936	3277360
A11	1708904	0.34	341841936	3277360	3.04	9128	3233827	1.01	1.14	0.59	1.0	341841936	3277360
A12	1612470	0.34	341841936	3277360	3.23	9680	2875432	1.07	1.22	0.55	1.0	341841936	3277360
O1-NL	1649141	0.34	423174646	3772585	2.33	11639	2644346	1.19	1.38	0.48	1.0	423174646	3772585
O2-NL	1519941	0.34	423174646	3772585	2.48	12421	2321877	1.27	1.50	0.44	1.0	423174646	3772585
O3-NL	1405168	0.34	423174646	3772585	2.63	13173	2064550	1.35	1.61	0.40	1.0	423174646	3772585
A1-NL	190076	0.34	40386575	855550	2.33	6984	246298	1.86	2.52	0.24	1.0	40386575	855550
A2-NL	170842	0.34	40386575	855550	2.48	7453	216263	1.99	2.78	0.21	1.0	40386575	855550
A3-NL	154800	0.34	40386575	855550	2.63	7904	192295	2.11	3.05	0.19	1.0	40386575	855550
A4-NL	462835	0.34	82663170	1307820	2.33	6984	677645	1.39	1.67	0.39	1.0	82663170	1307820
A5-NL	421980	0.34	82663170	1307820	2.48	7453	595008	1.48	1.82	0.35	1.0	82663170	1307820
A6-NL	386702	0.34	82663170	1307820	2.63	7904	529065	1.57	1.97	0.32	1.0	82663170	1307820
A7-NL	1180162	0.34	192918360	2185380	2.33	6984	2203124	1.00	1.13	0.60	1.0	192918360	2185380
A8-NL	1105774	0.34	192918360	2185380	2.48	7453	1934460	1.06	1.21	0.56	1.0	192918360	2185380
A9-NL	1036770	0.34	192918360	2185380	2.63	7904	1720069	1.13	1.29	0.52	1.0	192918360	2185380
A10-NL	2190255	0.34	341841936	3277360	2.33	6984	5524428	0.77	0.89	0.74	1.0	341841936	3277360
A11-NL	2101455	0.34	341841936	3277360	2.48	7453	4850741	0.82	0.94	0.71	1.0	341841936	3277360
A12-NL	2014564	0.34	341841936	3277360	2.63	7904	4313148	0.87	0.99	0.68	1.0	341841936	3277360
O1-V	357434	0.34	423174646	3772585	5.70	28511	440724	2.93	5.24	0.10	1.0	423174646	3772585
O2-V	319852	0.34	423174646	3772585	6.09	30426	386979	3.12	5.87	0.09	1.0	423174646	3772585
O3-V	288839	0.34	423174646	3772585	6.45	32267	344092	3.31	6.51	0.08	1.0	423174646	3772585
A1-V	35867	0.34	40386575	855550	5.70	17106	41050	4.57	11.66	0.04	1.0	40386575	855550
A2-V	31874	0.34	40386575	855550	6.09	18256	36044	4.87	13.16	0.04	1.0	40386575	855550
A3-V	28622	0.34	40386575	855550	6.45	19360	32049	5.17	14.69	0.04	1.0	40386575	855550
A4-V	94218	0.34	82663170	1307820	5.70	17106	112941	3.40	6.83	0.08	1.0	82663170	1307820
A5-V	84088	0.34	82663170	1307820	6.09	18256	99168	3.63	7.68	0.07	1.0	82663170	1307820
A6-V	75771	0.34	82663170	1307820	6.45	19360	88178	3.85	8.54	0.06	1.0	82663170	1307820
A7-V	284251	0.34	192918360	2185380	5.70	17106	367187	2.44	3.86	0.15	1.0	192918360	2185380
A8-V	255538	0.34	192918360	2185380	6.09	18256	322410	2.60	4.30	0.13	1.0	192918360	2185380
A9-V	231623	0.34	192918360	2185380	6.45	19360	286678	2.76	4.75	0.12	1.0	192918360	2185380
A10-V	647169	0.34	341841936	3277360	5.70	17106	920738	1.89	2.57	0.23	1.0	341841936	3277360
A11-V	587585	0.34	341841936	3277360	6.09	18256	808457	2.01	2.84	0.21	1.0	341841936	3277360
A12-V	536894	0.34	341841936	3277360	6.45	19360	718858	2.14	3.11	0.19	1.0	341841936	3277360
O1-H	1115881	0.34	423174646	3772585	2.85	14255	1762898	1.46	1.78	0.36	1.0	423174646	3772585
O2-H	972863	0.34	423174646	3772585	3.04	15213	1547918	1.56	1.95	0.32	1.0	423174646	3772585
O3-H	852730	0.34	423174646	3772585	3.23	16133	1376367	1.66	2.12	0.29	1.0	423174646	3772585
A1-H	110165	0.34	40386575	855550	2.85	8553	164199	2.28	3.46	0.17	1.0	40386575	855550
A2-H	89218	0.34	40386575	855550	3.04	9128	144175	2.44	3.85	0.15	1.0	40386575	855550
A3-H	72328	0.34	40386575	855550	3.23	9680	128197	2.58	4.24	0.13	1.0	40386575	855550
A4-H	303249	0.34	82663170	1307820	2.85	8553	451763	1.70	2.20	0.28	1.0	82663170	1307820
A5-H	261584	0.34	82663170	1307820	3.04	9128	396672	1.82	2.42	0.25	1.0	82663170	1307820
A6-H	227101	0.34	82663170	1307820	3.23	9680	352710	1.93	2.65	0.22	1.0	82663170	1307820
A7-H	871526	0.34	192918360	2185380	2.85	8553	1468749	1.22	1.42	0.47	1.0	192918360	2185380
A8-H	786796	0.34	192918360	2185380	3.04	9128	1289640	1.30	1.53	0.43	1.0	192918360	2185380
A9-H	712792	0.34	192918360	2185380	3.23	9680	1146713	1.38	1.65	0.39	1.0	192918360	2185380
A10-H	1767596	0.34	341841936	3277360	2.85	8553	3682952	0.94	1.07	0.63	1.0	341841936	3277360
A11-H	1646879	0.34	341841936	3277360	3.04	9128	3233827	1.01	1.14	0.59	1.0	341841936	3277360
A12-H	1534749	0.34	341841936	3277360	3.23	9680	2875432	1.07	1.22	0.55	1.0	341841936	3277360
O1-VH	331854	0.34	423174646	3772585	5.70	28511	440724	2.93	5.24	0.10	1.0	423174646	3772585
O2-VH	285835	0.34	423174646	3772585	6.09	30426	386979	3.12	5.87	0.09	1.0	423174646	3772585
O3-VH	248201	0.34	423174646	3772585	6.45	32267	344092	3.31	6.51	0.08	1.0	423174646	3772585
A1-VH	30300	0.34	40386575	855550	5.70	17106	41050	4.57	11.66	0.04	1.0	40386575	855550
A2-VH	24495	0.34	40386575	855550	6.09	18256	36044	4.87	13.16	0.04	1.0	40386575	855550
A3-VH	19829	0.34	40386575	855550	6.45	19360	32049	5.17	14.69	0.04	1.0	40386575	855550
A4-VH	86887	0.34	82663170	1307820	5.70	17106	112941	3.40	6.83	0.08	1.0	82663170	1307820
A5-VH	74349	0.34	82663170	1307820	6.09	18256	99168	3.63	7.68	0.07	1.0	82663170	1307820
A6-VH	64144	0.34	82663170	1307820	6.45	19360	88178	3.85	8.54	0.06	1.0	82663170	1307820
A7-VH	274337	0.34	192918360	2185380	5.70	17106	367187	2.44	3.86	0.15	1.0	192918360	2185380
A8-VH	242305	0.34	192918360	2185380	6.09	18256	322410	2.60	4.30	0.13	1.0	192918360	2185380
A9-VH	215770	0.34	192918360	2185380	6.45	19360	286678	2.76	4.75	0.12	1.0	192918360	2185380
A10-VH	633859	0.34	341841936	3277360	5.70	17106	920738	1.89	2.57	0.23	1.0	341841936	3277360
A11-VH	569656	0.34	341841936	3277360	6.09	18256	808457	2.01	2.84	0.21	1.0	341841936	3277360
A12-VH	515273	0.34	341841936	3277360	6.45	19360	718858	2.14	3.11	0.19	1.0	341841936	3277360
O1-NLH	1503290	0.34	423174646	3772585	2.33	11639	2644346	1.19	1.38	0.48	1.0	423174646	3772585
O2-NLH	1323252	0.34	423174646	3772585	2.48	12421	2321877	1.27	1.50	0.44	1.0	423174646	3772585
O3-NLH	1168495	0.34	423174646	3772585	2.63	13173	2064550	1.35	1.61	0.40	1.0	423174646	3772585
A1-NLH	156775	0.34	40386575	855550	2.33	6984	246298	1.86	2.52	0.24	1.0	40386575	855

Current EC1993-1-1 with Yura					
	$1/(1-((F_{ult}/F_{Rd})^{\alpha})^{\lambda^2})$		$(2^{\alpha} \cdot n_{Ed} + (Q/F) \cdot \rho_{NEd} + H) \cdot h \cdot C$	$F/F_{Rd} + (n/n-1) \cdot F_{10} \cdot e^2 / M_{Rd}$	
k	ϕF	C_m	M_{Ed}	SOLVE FOR = 1, F_{ult}	
	N	[-]	Nmm	[-]	
O1	1.33	4446.0	0.9	33344800	1.00
O2	1.30	4047.3	0.9	30354405	1.00
O3	1.27	3704.6	0.9	27784570	1.00
A1	1.15	5405.0	0.9	2432081	1.00
A2	1.14	483.6	0.9	2176016	1.00
A3	1.12	436.6	0.9	1964652	1.00
A4	1.26	1358.4	0.9	6112805	1.00
A5	1.23	1227.4	0.9	5523264	1.00
A6	1.21	1116.8	0.9	5025782	1.00
A7	1.41	3713.3	0.9	16709717	1.00
A8	1.38	3423.2	0.9	15404369	1.00
A9	1.36	3165.8	0.9	14245930	1.00
A10	1.45	7395.0	0.9	33277717	1.00
A11	1.46	6976.6	0.9	31394570	1.00
A12	1.45	6582.9	0.9	29622974	1.00
O1-NL	1.43	6021.8	0.9	30109052	1.00
O2-NL	1.40	5550.0	0.9	27750199	1.00
O3-NL	1.38	5130.9	0.9	25654742	1.00
A1-NL	1.22	776.0	0.9	2327948	1.00
A2-NL	1.20	697.5	0.9	2092375	1.00
A3-NL	1.18	632.0	0.9	1895906	1.00
A4-NL	1.36	1889.5	0.9	5668549	1.00
A5-NL	1.33	1722.7	0.9	5168175	1.00
A6-NL	1.30	1578.7	0.9	4736115	1.00
A7-NL	1.47	4818.0	0.9	14453974	1.00
A8-NL	1.47	4514.3	0.9	13542905	1.00
A9-NL	1.46	4232.6	0.9	12697786	1.00
A10-NL	1.42	8941.7	0.9	26825040	1.00
A11-NL	1.44	8579.2	0.9	25737457	1.00
A12-NL	1.46	8224.4	0.9	24673269	1.00
O1-V	1.09	1305.2	0.9	39154958	1.00
O2-V	1.08	1167.9	0.9	35038056	1.00
O3-V	1.07	1054.7	0.9	31640741	1.00
A1-V	1.04	146.4	0.9	2635688	1.00
A2-V	1.04	130.1	0.9	2342255	1.00
A3-V	1.03	116.8	0.9	2103263	1.00
A4-V	1.07	384.6	0.9	6923574	1.00
A5-V	1.06	343.3	0.9	6179157	1.00
A6-V	1.06	309.3	0.9	5568023	1.00
A7-V	1.13	1160.5	0.9	20888104	1.00
A8-V	1.11	1043.2	0.9	18778161	1.00
A9-V	1.10	945.6	0.9	17020716	1.00
A10-V	1.20	2642.1	0.9	47557019	1.00
A11-V	1.18	2398.8	0.9	43178475	1.00
A12-V	1.16	2191.9	0.9	39453518	1.00
O1-H	1.29	4074.6	0.9	61809662	1.00
O2-H	1.25	3524.2	0.9	73517957	1.00
O3-H	1.22	3113.7	0.9	85852972	1.00
A1-H	1.12	449.7	0.9	8773864	1.00
A2-H	1.10	364.2	0.9	11764033	1.00
A3-H	1.08	295.3	0.9	14828755	1.00
A4-H	1.23	1238.0	0.9	12321046	1.00
A5-H	1.20	1067.9	0.9	14930604	1.00
A6-H	1.17	927.1	0.9	17672105	1.00
A7-H	1.38	3558.0	0.9	22760958	1.00
A8-H	1.35	3212.1	0.9	24579361	1.00
A9-H	1.32	2910.0	0.9	26594829	1.00
A10-H	1.44	7216.2	0.9	39222815	1.00
A11-H	1.43	6723.4	0.9	40380103	1.00
A12-H	1.42	6265.6	0.9	41695135	1.00
O1-VH	1.09	1211.8	0.9	67602732	1.00
O2-VH	1.07	1043.7	0.9	78186650	1.00
O3-VH	1.06	906.3	0.9	89689093	1.00
A1-VH	1.03	123.7	0.9	8976565	1.00
A2-VH	1.03	100.0	0.9	11925039	1.00
A3-VH	1.02	81.0	0.9	14957138	1.00
A4-VH	1.06	354.7	0.9	13134898	1.00
A5-VH	1.05	303.5	0.9	15588490	1.00
A6-VH	1.05	261.9	0.9	18213614	1.00
A7-VH	1.12	1120.0	0.9	26909536	1.00
A8-VH	1.11	989.2	0.9	27930717	1.00
A9-VH	1.10	880.9	0.9	29355825	1.00
A10-VH	1.19	2587.7	0.9	53328899	1.00
A11-VH	1.17	2325.6	0.9	51986032	1.00
A12-VH	1.15	2103.6	0.9	51364661	1.00
O1-NLH	1.38	5489.2	0.9	58696203	1.00
O2-NLH	1.33	4831.8	0.9	71034163	1.00
O3-NLH	1.30	4266.7	0.9	83833697	1.00
A1-NLH	1.18	640.0	0.9	8670099	1.00
A2-NLH	1.14	519.2	0.9	11682459	1.00
A3-NLH	1.11	421.4	0.9	14764182	1.00
A4-NLH	1.31	1711.7	0.9	11885167	1.00
A5-NLH	1.27	1486.1	0.9	14583330	1.00
A6-NLH	1.23	1296.6	0.9	17389764	1.00
A7-NLH	1.44	4608.2	0.9	20574710	1.00
A8-NLH	1.42	4222.0	0.9	22791000	1.00
A9-NLH	1.40	3871.6	0.9	25114814	1.00
A10-NLH	1.40	8727.3	0.9	32932037	1.00
A11-NLH	1.42	8266.9	0.9	34925601	1.00
A12-NLH	1.43	7822.3	0.9	36966888	1.00

Current EC1993-1-1 with L.F. Geschwindner									
variable for solve function		$W \cdot F_{yd}$	$A \cdot F_{yd}$		$(E_L/L_c) / ((E_L/L_n))$		eq=0		
F_{ult}	α	M_{Rk}	N_{Rk}	G_A	G_B	K	SOLVE FOR = 0, K		
N	[-]	Nmm	N	[-]	[-]	[-]	[-]	[-]	
1383838	0.34	423174646	3772585	1E+104	0.500	2.59	0.00		
1311735	0.34	423174646	3772585	1E+104	0.750	2.70	0.00		
1245353	0.34	423174646	3772585	1E+104	1.000	2.81	0.00		
154962	0.34	40386575	855550	1E+104	0.500	2.59	0.00		
144950	0.34	40386575	855550	1E+104	0.750	2.70	0.00		
136036	0.34	40386575	855550	1E+104	1.000	2.81	0.00		
382971	0.34	82663170	1307820	1E+104	0.500	2.59	0.00		
360968	0.34	82663170	1307820	1E+104	0.750	2.70	0.00		
341009	0.34	82663170	1307820	1E+104	1.000	2.81	0.00		
1012864	0.34	192918360	2185380	1E+104	0.500	2.59	0.00		
968789	0.34	192918360	2185380	1E+104	0.750	2.70	0.00		
927230	0.34	192918360	2185380	1E+104	1.000	2.81	0.00		
1949912	0.34	341841936	3277360	1E+104	0.500	2.59	0.00		
1892245	0.34	341841936	3277360	1E+104	0.750	2.70	0.00		
1835943	0.34	341841936	3277360	1E+104	1.000	2.81	0.00		
1792646	0.34	423174646	3772585	1E+104	0.500	2.17	0.00		
1719189	0.34	423174646	3772585	1E+104	0.750	2.25	0.00		
1649141	0.34	423174646	3772585	1E+104	1.000	2.33	0.00		
213212	0.34	40386575	855550	1E+104	0.500	2.17	0.00		
201113	0.34	40386575	855550	1E+104	0.750	2.25	0.00		
190076	0.34	40386575	855550	1E+104	1.000	2.33	0.00		
509822	0.34	82663170	1307820	1E+104	0.500	2.17	0.00		
485546	0.34	82663170	1307820	1E+104	0.750	2.25	0.00		
462835	0.34	82663170	1307820	1E+104	1.000	2.33	0.00		
1258887	0.34	192918360	2185380	1E+104	0.500	2.17	0.00		
1219086	0.34	192918360	2185380	1E+104	0.750	2.25	0.00		
1180162	0.34	192918360	2185380	1E+104	1.000	2.33	0.00		
2279637	0.34	341841936	3277360	1E+104	0.500	2.17	0.00		
2234978	0.34	341841936	3277360	1E+104	0.750	2.25	0.00		
2190255	0.34	341841936	3277360	1E+104	1.000	2.33	0.00		
441385	0.34	423174646	3772585	1E+104	0.500	5.02	0.00		
409532	0.34	423174646	3772585	1E+104	0.750	5.26	0.00		
381882	0.34	423174646	3772585	1E+104	1.000	5.48	0.00		
45008	0.34	40386575	855550	1E+104	0.500	5.02	0.00		
41502	0.34	40386575	855550	1E+104	0.750	5.26	0.00		
39497	0.34	40386575	855550	1E+104	1.000	5.48	0.00		
117067	0.34	82663170	1307820	1E+104	0.500	5.02	0.00		
108360	0.34	82663170	1307820	1E+104	0.750	5.26	0.00		
100840	0.34	82663170	1307820	1E+104	1.000	5.48	0.00		
347283	0.34	192918360	2185380	1E+104	0.500	5.02	0.00		
323551	0.34	192918360	2185380	1E+104	0.750	5.26	0.00		
302768	0.34	192918360	2185380	1E+104	1.000	5.48	0.00		
773006	0.34	341841936	3277360	1E+104	0.500	5.02	0.00		
726435	0.34	341841936	3277360	1E+104	0.750	5.26	0.00		
684848	0.34	341841936	3277360	1E+104	1.000	5.48	0.00		
1266206	0.34	423174646	3772585	1E+104	0.500	2.59	0.00		
1147619	0.34	423174646	3772585	1E+104	0.750	2.70	0.00		
1041287	0.34	423174646	3772585	1E+104	1.000	2.81	0.00		
128552	0.34	40386575	855550	1E+104	0.500	2.59	0.00		
108588	0.34	40386575	855550	1E+104	0.750	2.70	0.00		
91280	0.34	40386575	855550	1E+104	1.000	2.81	0.00		
348355	0.34	82663170	1307820	1E+104	0.500	2.59	0.00		
312932	0.34	82663170	1307820	1E+104	0.750	2.70	0.00		
281528	0.34	82663170	1307820	1E+104	1.000	2.81	0.00		
970140	0.34	192918360	2185380	1E+104	0.500	2.59	0.00		
908095	0.34	192918360	2185380	1E+104	0.750	2.70	0.00		
850600	0.34	192918360	2185380	1E+104	1.000	2.81	0.00		
1903425	0.34	341841936	3277360	1E+104	0.500	2.59	0.00		
1824380	0.34	341841936	3277360	1E+104	0.750	2.70	0.00		
1748063	0.34	341841936	3277360	1E+104	1.000	2.81	0.00		
409350	0.34	423174646	3772585	1E+104	0.500	5.02	0.00		
365324	0.34	423174646	3772585	1E+104	0.750	5.26	0.00		
327330	0.34	423174646	3772585	1E+104	1.000	5.48	0.00</		

Current EC1993-1-1 with L.F. Geschwindner													
k*h	$\pi^2 EI / L_{cr}^2$	$\lambda = \sqrt{(F_{Rk} / F_{cr})}$	$\Phi = 0.5(1 + \alpha + (\lambda - 0.2) + \lambda^2)$	$\chi = 1 / (\Phi + \sqrt{(\Phi^2 - \lambda^2)})$	M_{Rk} / Y_{M0}	N_{Rk} / Y_{M0}	$1 / (1 - ((F_{ult} / F_{Rd}) * \chi^2 * \lambda^2))$		$(2 * \phi N_{Ed} * (Q / F) + \phi N_{Ed} + H) * h * C$	$F / N_{Rd} + (n / n - 1) * F_{tot} * e' / M_{Rd}$			
L_{cr}	N_{cr}	λ	Φ	χ	Y_{M0}	M_{Rd}	N_{Rd}	k	ϕF	c_m	M_{Ed}	SOLVE FOR = 1, F_{ult}	
mm	N	[-]	[-]	[-]	[-]	Nmm	N	[-]	N	[-]	Nmm	[-]	
01	12971	2129248	1.33	1.58	0.41	1.0	423174646	3772585	1.37	5053.1	0.9	37897973	1.00
02	13506	1963878	1.39	1.66	0.39	1.0	423174646	3772585	1.35	4789.8	0.9	35923344	1.00
03	14028	1820560	1.44	1.75	0.37	1.0	423174646	3772585	1.33	4547.4	0.9	34105407	1.00
A1	7783	198321	2.08	2.98	0.20	1.0	40386575	855550	1.18	632.6	0.9	2846832	1.00
A2	8104	182918	2.16	3.17	0.18	1.0	40386575	855550	1.17	591.8	0.9	2662910	1.00
A3	8417	169570	2.25	3.37	0.17	1.0	40386575	855550	1.16	555.4	0.9	2499141	1.00
A4	7783	545645	1.55	1.93	0.33	1.0	82663170	1307820	1.30	1563.5	0.9	7035631	1.00
A5	8104	503267	1.61	2.04	0.30	1.0	82663170	1307820	1.28	1473.6	0.9	6631408	1.00
A6	8417	466540	1.67	2.15	0.29	1.0	82663170	1307820	1.26	1392.2	0.9	6264743	1.00
A7	7783	1773972	1.11	1.27	0.53	1.0	192918360	2185380	1.43	4135.0	0.9	18607498	1.00
A8	8104	1636195	1.16	1.33	0.50	1.0	192918360	2185380	1.42	3955.1	0.9	17797791	1.00
A9	8417	1516791	1.20	1.39	0.48	1.0	192918360	2185380	1.41	3785.4	0.9	17034296	1.00
A10	7783	4448311	0.86	0.98	0.69	1.0	341841936	3277360	1.43	7960.5	0.9	35822176	1.00
A11	8104	4102830	0.89	1.02	0.67	1.0	341841936	3277360	1.44	7725.1	0.9	34762758	1.00
A12	8417	3803418	0.93	1.05	0.64	1.0	341841936	3277360	1.45	7495.2	0.9	33728435	1.00
01-NL	10829	3054758	1.11	1.27	0.53	1.0	423174646	3772585	1.45	6545.8	0.9	32729081	1.00
02-NL	11238	2836863	1.15	1.33	0.50	1.0	423174646	3772585	1.44	6277.6	0.9	31387951	1.00
03-NL	11639	2644346	1.19	1.38	0.48	1.0	423174646	3772585	1.43	6021.8	0.9	30109052	1.00
A1-NL	6498	284524	1.73	2.26	0.27	1.0	40386575	855550	1.25	870.4	0.9	2611307	1.00
A2-NL	6743	264229	1.80	2.39	0.25	1.0	40386575	855550	1.24	821.0	0.9	2463116	1.00
A3-NL	6984	246298	1.86	2.52	0.24	1.0	40386575	855550	1.22	776.0	0.9	2327948	1.00
A4-NL	6498	782817	1.29	1.52	0.43	1.0	82663170	1307820	1.39	2081.3	0.9	6244025	1.00
A5-NL	6743	726979	1.34	1.59	0.41	1.0	82663170	1307820	1.37	1982.2	0.9	5946704	1.00
A6-NL	6984	677645	1.39	1.67	0.39	1.0	82663170	1307820	1.36	1889.5	0.9	5668549	1.00
A7-NL	6498	2545056	0.93	1.05	0.64	1.0	192918360	2185380	1.47	5139.4	0.9	15418154	1.00
A8-NL	6743	2363518	0.96	1.09	0.62	1.0	192918360	2185380	1.47	4976.9	0.9	14930688	1.00
A9-NL	6984	2203124	1.00	1.13	0.60	1.0	192918360	2185380	1.47	4818.0	0.9	14453974	1.00
A10-NL	6498	6381837	0.72	0.84	0.77	1.0	341841936	3277360	1.38	9306.6	0.9	27919739	1.00
A11-NL	6743	5926623	0.74	0.87	0.76	1.0	341841936	3277360	1.40	9124.3	0.9	27372774	1.00
A12-NL	6984	5524428	0.77	0.89	0.74	1.0	341841936	3277360	1.42	8941.7	0.9	26825040	1.00
01-V	25104	568433	2.58	4.22	0.13	1.0	423174646	3772585	1.11	1611.7	0.9	48351267	1.00
02-V	26281	518666	2.70	4.56	0.12	1.0	423174646	3772585	1.11	1495.4	0.9	44861964	1.00
03-V	27411	476785	2.81	4.90	0.11	1.0	423174646	3772585	1.10	1394.4	0.9	41833129	1.00
A1-V	15063	52945	4.02	9.23	0.06	1.0	40386575	855550	1.05	183.7	0.9	3307366	1.00
A2-V	15769	48309	4.21	10.04	0.05	1.0	40386575	855550	1.05	169.4	0.9	3049768	1.00
A3-V	16447	44408	4.39	10.84	0.05	1.0	40386575	855550	1.04	157.2	0.9	2828919	1.00
A4-V	15063	145668	3.00	5.46	0.10	1.0	82663170	1307820	1.09	477.9	0.9	8602625	1.00
A5-V	15769	132914	3.14	5.92	0.09	1.0	82663170	1307820	1.08	442.4	0.9	7962822	1.00
A6-V	16447	122182	3.27	6.37	0.08	1.0	82663170	1307820	1.07	411.7	0.9	7410185	1.00
A7-V	15063	473587	2.15	3.14	0.18	1.0	192918360	2185380	1.16	1417.8	0.9	25520020	1.00
A8-V	15769	432124	2.25	3.38	0.17	1.0	192918360	2185380	1.15	1320.9	0.9	23776063	1.00
A9-V	16447	397231	2.35	3.62	0.16	1.0	192918360	2185380	1.14	1236.0	0.9	22248784	1.00
A10-V	15063	1187540	1.66	2.13	0.29	1.0	341841936	3277360	1.23	3155.8	0.9	56804112	1.00
A11-V	15769	1083568	1.74	2.27	0.27	1.0	341841936	3277360	1.22	2965.7	0.9	53381844	1.00
A12-V	16447	996074	1.81	2.42	0.25	1.0	341841936	3277360	1.21	2795.9	0.9	50325811	1.00
01-H	12971	2129248	1.33	1.58	0.41	1.0	423174646	3772585	1.32	4623.5	0.9	65926482	1.00
02-H	13506	1963878	1.39	1.66	0.39	1.0	423174646	3772585	1.29	4190.5	0.9	78303828	1.00
03-H	14028	1820560	1.44	1.75	0.37	1.0	423174646	3772585	1.26	3802.2	0.9	91016819	1.00
A1-H	7783	198321	2.08	2.98	0.20	1.0	40386575	855550	1.15	524.8	0.9	9111644	1.00
A2-H	8104	182918	2.16	3.17	0.18	1.0	40386575	855550	1.12	443.3	0.9	12119885	1.00
A3-H	8417	169570	2.25	3.37	0.17	1.0	40386575	855550	1.10	372.6	0.9	115176917	1.00
A4-H	7783	545645	1.55	1.93	0.33	1.0	82663170	1307820	1.26	1422.2	0.9	13149684	1.00
A5-H	8104	503267	1.61	2.04	0.30	1.0	82663170	1307820	1.23	1277.5	0.9	15873933	1.00
A6-H	8417	466540	1.67	2.15	0.29	1.0	82663170	1307820	1.21	1149.3	0.9	18672002	1.00
A7-H	7783	1773972	1.11	1.27	0.53	1.0	192918360	2185380	1.41	3960.6	0.9	24572608	1.00
A8-H	8104	1636195	1.16	1.33	0.50	1.0	192918360	2185380	1.39	3707.3	0.9	26807779	1.00
A9-H	8417	1516791	1.20	1.39	0.48	1.0	192918360	2185380	1.37	3472.6	0.9	29126514	1.00
A10-H	7783	4448311	0.86	0.98	0.69	1.0	341841936	3277360	1.42	7770.7	0.9	41718144	1.00
A11-H	8104	4102830	0.89	1.02	0.67	1.0	341841936	3277360	1.42	7448.0	0.9	43640998	1.00
A12-H	8417	3803418	0.93	1.05	0.64	1.0	341841936	3277360	1.42	7136.4	0.9	45613964	1.00
01-VH	25104	568433	2.58	4.22	0.13	1.0	423174646	3772585	1.11	1494.7	0.9	76092083	1.00
02-VH	26281	518666	2.70	4.56	0.12	1.0	423174646	3772585	1.09	1334.0	0.9	86894268	1.00
03-VH	27411	476785	2.81	4.90	0.11	1.0	423174646	3772585	1.08	1195.2	0.9	98357213	1.00
A1-VH	15063	52945	4.02	9.23	0.06	1.0	40386575	855550	1.04	155.0	0.9	9540362	1.00
A2-VH	15769	48309	4.21	10.04	0.05	1.0	40386575	855550	1.04	129.9	0.9	12463924	1.00
A3-VH	16447	44408	4.39	10.84	0.05	1.0	40386575	855550	1.03	108.6	0.9	15454291	1.00
A4-VH	15063	145668	3.00	5.46	0.10	1.0	82663170	1307820	1.08	440.3	0.9	14675535	1.00
A5-VH	15769	132914	3.14	5.92	0.09	1.0	82663170	1307820	1.07	390.5	0.9	17154365	1.00
A6-VH	16447	122182	3.27	6.37	0.08	1.0	82663170	1307820	1.06	347.7	0.9	19759150	1.00
A7-VH	15063	473587	2.15	3.14	0.18	1.0	192918360	2185380	1.15	1367.5	0.9	31364632	1.00
A8-VH	15769	432124	2.25	3.38	0.17	1.0	192918360	2185380	1.14	1251.2	0.9	32646240	1.00
A9-VH	16447	397231	2.35	3.62	0.16	1.0	192918360	2185380	1.13	1149.7	0.9	34194934	1.00
A10-VH	15063	1187540	1.66	2.13	0.29	1.0	341841936	3277360	1.23	3089.8	0.9	62366034	1.00
A11-VH	15769	1083568	1.74	2.27	0.27	1.0	341841936	3277360	1.21	2873.3	0.9	61844515	1.00
A12-VH	16447	996074	1.81	2.42	0.25	1.0	341841936	3277360	1.20	2680.6	0.9	61751335	1.00
01-NLH	10829	3054758	1.11	1.27	0.53	1.0	423174646	3772585	1.39	5962.6	0.9	61063181	1.00
02-NLH	11238	2836863	1.15	1.33	0.50	1.0	423174646	3772585	1.36	5454.2	0.9	74145953	1.00
03-NLH	11639	2644346	1.19	1.38	0.48	1.0	423174646	3772585	1.33	4987.9	0.9	87439488	1.00
A1-NLH	6498	284524	1.73	2.26	0.27	1.0	40386575	855550	1.20	715.7	0.9	8897249	1.00
A2-NLH	6743	264229	1.80	2.39	0.25	1.0	40386575	855550	1.17	607.5	0.9	11947556	1.00
A3-NLH	6984	246298	1.86	2.52	0.24	1.0	40386575	855550	1.14	512.7	0.9	15038129	1.00
A4-NLH	6498	782817	1.29										

NEN 6771 with Nomogram															
variable for solve function			$(EI_c/L_c)/(\mu(EI_b/L_b))$		$C_A+C_B*\lambda^2*\sin(\lambda)$	$(C_A+C_B)*\lambda*\cos(\lambda)+\sin(\lambda)$	eq1-eq2=0	$\pi/\lambda (K_0)$	$\pi/\lambda^3 h$	$\sqrt{I(A)}$	$\pi*\sqrt{E(f_{y0})}$	L_{cr}/l	λ/λ_e	$Wt*f_{y0}$	
F_{ult}	μ	C_A	C_B	λ	eq1	eq2	SOLVE FOR = 0, λ	nomogram	l_{cr}	i	λ_e	λ	λ_{rel}	M_{Rk}	
N	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	mm	mm	mm	mm	[-]	Nmm	
B01	2633298	6	0	0.167	2.72	0	1.43047E-09	0.00	1.157	5783	127.53	76.41	45.34	0.59	423174646.4
B02	2555024	6	0	0.250	2.57	0	1.55102E-10	0.00	1.222	6111	127.53	76.41	47.92	0.63	423174646.4
B03	2487218	6	0	0.333	2.46	0	-7.36087E-11	0.00	1.279	6397	127.53	76.41	50.16	0.66	423174646.4
B1	428014	6	0	0.167	2.72	0	1.43047E-09	0.00	1.157	3470	49.04	76.41	70.75	0.93	40386575
B2	405255	6	0	0.250	2.57	0	1.55102E-10	0.00	1.222	3667	49.04	76.41	74.77	0.98	40386575
B3	386321	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.279	3838	49.04	76.41	78.26	1.02	40386575
B4	831361	6	0	0.167	2.72	0	1.43047E-09	0.00	1.157	3470	65.79	76.41	52.74	0.69	82663170
B5	800801	6	0	0.250	2.57	0	1.55102E-10	0.00	1.222	3667	65.79	76.41	55.73	0.73	82663170
B6	774584	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.279	3838	65.79	76.41	58.34	0.76	82663170
B7	1660176	6	0	0.167	2.72	0	1.43047E-09	0.00	1.157	3470	91.77	76.41	37.81	0.49	192918360
B8	1621783	6	0	0.250	2.57	0	1.55102E-10	0.00	1.222	3667	91.77	76.41	39.96	0.52	192918360
B9	1588324	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.279	3838	91.77	76.41	41.82	0.55	192918360
B10	2710193	6	0	0.167	2.72	0	1.43047E-09	0.00	1.157	3470	118.66	76.41	29.24	0.38	341841936.4
B11	2664910	6	0	0.250	2.57	0	1.55102E-10	0.00	1.222	3667	118.66	76.41	30.90	0.40	341841936.4
B12	2625295	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.279	3838	118.66	76.41	32.34	0.42	341841936.4

NEN 6771 with Nomogram																
$A*f_{y0}$			$\alpha_k*(\lambda_{rel}*\lambda_0)*M_{Rk}/N_{Rk}$		M_{Rd}/γ_{M0}	N_{Rk}/γ_{M0}	$\pi^2 EI_c/L_{cr}^2$	N_{cr}/F_{ult}			factor* N_{Ed}	$\phi_0\alpha_k\alpha_x\alpha_m$		boundary factor	$F_{ult}*h^3/2*C_m*\phi$	$F/N_{Rd}*(n/n-1)*F_{ult}*e^*/M_{Rd}$
N_{Rk}	α_k	λ_0	e^*	γ_{M0}	M_{Rd}	N_{Rd}	N_{cr}	n	N_{Ed}	factor	F_{tot}	ϕ	ϕF	bf	M_{Ed}	SOLVE FOR = 1, F_{ult}
N	[-]	[-]	mm	[-]	Nmm	N	N	[-]	N	[-]	N	[-]	N	[-]	Nmm	[-]
B01	3772585	0.34	0.20	15.00	1	423174646.4	3772585	10713839	4.07	2633297.8	1.5	3949946.7	0.00365	9615.4	0.57	37121854.4
B02	3772585	0.34	0.20	16.29	1	423174646.4	3772585	9592917	3.75	2555024.3	1.5	3832536.5	0.00365	9329.6	0.60	37769970.2
B03	3772585	0.34	0.20	17.41	1	423174646.4	3772585	8755266	3.52	2487218.4	1.5	3730827.5	0.00365	9082.0	0.62	38278389.6
B1	855550	0.34	0.20	11.65	1	40386575	855550	997902	2.33	428014.4	1.5	642021.6	0.00408	1747.4	0.57	4045405.5
B2	855550	0.34	0.20	12.50	1	40386575	855550	893498	2.20	405254.6	1.5	607881.9	0.00408	1654.4	0.60	4019646.2
B3	855550	0.34	0.20	13.23	1	40386575	855550	815478	2.11	386320.9	1.5	579481.4	0.00408	1577.1	0.62	3990595.9
B4	1307820	0.34	0.20	10.53	1	82663170	1307820	2745547	3.30	831361.2	1.5	1247041.7	0.00408	3394.0	0.57	7860015.6
B5	1307820	0.34	0.20	11.38	1	82663170	1307820	2458297	3.07	800800.7	1.5	1201201.1	0.00408	3269.3	0.60	7941965.7
B6	1307820	0.34	0.20	12.11	1	82663170	1307820	2243640	2.90	774583.6	1.5	1161875.4	0.00408	3162.2	0.62	7998745.2
B7	2185380	0.34	0.20	8.85	1	192918360	2185380	8926181	5.38	1660176.0	1.5	2490264.0	0.00408	6777.6	0.57	15705900.1
B8	2185380	0.34	0.20	9.69	1	192918360	2185380	7992290	4.93	1621783.0	1.5	2432674.6	0.00408	6620.9	0.60	16079614.2
B9	2185380	0.34	0.20	10.43	1	192918360	2185380	7294405	4.59	1588323.9	1.5	2382485.8	0.00408	6484.3	0.62	16390987.4
B10	3277360	0.34	0.20	6.48	1	341841936.4	3277360	22382783	8.26	2710193.2	1.5	4065289.8	0.00408	11064.3	0.57	25661869.8
B11	3277360	0.34	0.20	7.25	1	341841936.4	3277360	20041011	7.52	2664910.0	1.5	3997365.0	0.00408	10879.4	0.60	26411995.9
B12	3277360	0.34	0.20	7.92	1	341841936.4	3277360	18291036	6.97	2625294.6	1.5	3937941.8	0.00408	10717.7	0.62	27067495.0

Modified EC1993-1-1 with Nomogram																
variable for solve function		$W_i \cdot f_{yd}$	$A \cdot f_{yd}$			$(E I_x / L_x) / (\mu (E I_y / L_y))$		$C_A \cdot C_B \cdot \lambda^2 \cdot \sin(\lambda)$	$(C_A + C_B) \cdot \lambda \cdot \cos(\lambda) + \sin(\lambda)$	eq1-eq2=0	$\pi / \lambda (K_0)$	$\pi / \lambda \cdot h$	$\pi^2 E I_x / L_{cr}^2$	$\lambda = \sqrt{(F_{Rd} / F_{cr})}$	$\Phi = 0.5(1 + \alpha \cdot (\lambda - 0.2) + \lambda^2)$	
F_{ult}	α	M_{Rk}	N_{Rk}	μ	C_A	C_B	λ	eq1	eq2	SOLVE FOR = 0, λ	nomogram	L_{cr}	N_{cr}	λ'	Φ	
N	[-]	Nmm	N	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	mm	N	[-]	[-]	
801	2633298	0.34	423174646	3772585	6	0	0.167	2.72	0	1.43047E-09	0.00	1.16	5783	10713839	0.59	0.74
802	2555024	0.34	423174646	3772585	6	0	0.250	2.57	0	1.55102E-10	0.00	1.22	6111	9592917	0.63	0.77
803	2487218	0.34	423174646	3772585	6	0	0.333	2.46	0	-7.36087E-11	0.00	1.28	6397	8755266	0.66	0.79
81	428014	0.34	40386575	855550	6	0	0.167	2.72	0	1.43047E-09	0.00	1.16	3470	997902	0.93	1.05
82	405255	0.34	40386575	855550	6	0	0.250	2.57	0	1.55102E-10	0.00	1.22	3667	893498	0.98	1.11
83	386321	0.34	40386575	855550	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.28	3838	815478	1.02	1.16
84	831361	0.34	82663170	1307820	6	0	0.167	2.72	0	1.43047E-09	0.00	1.16	3470	2745547	0.69	0.82
85	800801	0.34	82663170	1307820	6	0	0.250	2.57	0	1.55102E-10	0.00	1.22	3667	2458297	0.73	0.86
86	774584	0.34	82663170	1307820	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.28	3838	2243640	0.76	0.89
87	1660176	0.34	192918360	2185380	6	0	0.167	2.72	0	1.43047E-09	0.00	1.16	3470	8926181	0.49	0.67
88	1621783	0.34	192918360	2185380	6	0	0.250	2.57	0	1.55102E-10	0.00	1.22	3667	7992290	0.52	0.69
89	1588324	0.34	192918360	2185380	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.28	3838	7294405	0.55	0.71
810	2710193	0.34	341841936	3277360	6	0	0.167	2.72	0	1.43047E-09	0.00	1.16	3470	22382783	0.38	0.60
811	2664910	0.34	341841936	3277360	6	0	0.250	2.57	0	1.55102E-10	0.00	1.22	3667	20041011	0.40	0.62
812	2625295	0.34	341841936	3277360	6	0	0.333	2.46	0	-7.36086E-11	0.00	1.28	3838	18291036	0.42	0.63

Modified EC1993-1-1 with Nomogram													
$\chi = 1 / (\Phi + \sqrt{(\Phi^2 - \lambda^2)})$		M_{Rd} / γ_{M0}	N_{Rd} / γ_{M0}		$1 / (1 - ((F_{ult} / F_{Rd}) \cdot \chi^2 \cdot \lambda^2))$			factor * N_{Ed}		boundary factor	$F_{ult} \cdot h^3 / 3 / 2 \cdot C_m \cdot b f$	$F / N_{Ed} + (n/n-1) \cdot F_{ult} \cdot e / M_{Rd}$	SOLVE FOR = 1, F_{ult}
χ	γ_{M0}	M_{Rd}	N_{Rd}	k	N_{Ed}	factor	F_{min}	φF	C_m	bf	M_{Ed}	N	[-]
[-]	[-]	Nmm	N	[-]	N	[-]	N	N	[-]	[-]	N	N	[-]
801	0.84	1.0	423174646	3772585	1.26	2633298	0.5	1316649	9615.4	0.9	0.57	41246505	1.00
802	0.82	1.0	423174646	3772585	1.28	2555024	0.5	1277512	9329.6	0.9	0.60	41966633	1.00
803	0.81	1.0	423174646	3772585	1.30	2487218	0.5	1243609	9082.0	0.9	0.62	42531544	1.00
81	0.64	1.0	40386575	855550	1.38	428014	0.5	214007	1747.4	0.9	0.57	4494895	1.00
82	0.61	1.0	40386575	855550	1.38	405255	0.5	202627	1654.4	0.9	0.60	4466274	1.00
83	0.58	1.0	40386575	855550	1.38	386321	0.5	193160	1577.1	0.9	0.62	4433995	1.00
84	0.79	1.0	82663170	1307820	1.31	831361	0.5	415681	3394.0	0.9	0.57	8733351	1.00
85	0.77	1.0	82663170	1307820	1.33	800801	0.5	400400	3269.3	0.9	0.60	8824406	1.00
86	0.75	1.0	82663170	1307820	1.35	774584	0.5	387292	3162.2	0.9	0.62	8887495	1.00
87	0.89	1.0	192918360	2185380	1.20	1660176	0.5	830088	6777.6	0.9	0.57	17451000	1.00
88	0.87	1.0	192918360	2185380	1.22	1621783	0.5	810892	6620.9	0.9	0.60	17866238	1.00
89	0.86	1.0	192918360	2185380	1.23	1588324	0.5	794162	6484.3	0.9	0.62	18212208	1.00
810	0.93	1.0	341841936	3277360	1.13	2710193	0.5	1355097	11064.3	0.9	0.57	28513189	1.00
811	0.92	1.0	341841936	3277360	1.14	2664910	0.5	1332455	10879.4	0.9	0.60	29346662	1.00
812	0.92	1.0	341841936	3277360	1.15	2625295	0.5	1312647	10717.7	0.9	0.62	30074995	1.00

Current EC1993-1-1 with LBA													
variable for solve function	$W_i^*f_{yd}$	A^*f_{yd}	LBA	$\lambda = \sqrt{(N_{Rk}/N_{cr})}$	$\Phi = 0.5(1+\alpha^*(\lambda-0.2))+\lambda^2$	$\chi = 1/(\Phi+\sqrt{(\Phi^2-\lambda^2)})$	M_{Rk}/Y_{M0}	N_{Rk}/Y_{M0}	$1/(1-(F_{Ed}/F_{Rd})\chi^{\lambda^2})$				
F_{Ed}	α	M_{Rk}	N_{Rk}	N_{cr}	λ	Φ	χ	Y_{M0}	M_{Rd}	N_{Rd}	k		N_{Ed}
N	[-]	Nmm	N	N	[-]	[-]	[-]	[-]	Nmm	N	[-]		N
B01	2548750	0.34	423174646	3772585	7045600	0.73	0.86	1.0	423174646	3772585	1.38		2548750
B02	2466145	0.34	423174646	3772585	6384900	0.77	0.89	1.0	423174646	3772585	1.40		2466145
B03	2391908	0.34	423174646	3772585	5866100	0.80	0.92	1.0	423174646	3772585	1.42		2391908
B1	389233	0.34	40386575	855550	681020	1.12	1.28	1.0	40386575	855550	1.43		389233
B2	364768	0.34	40386575	855550	612360	1.18	1.37	1.0	40386575	855550	1.41		364768
B3	344642	0.34	40386575	855550	560490	1.24	1.44	1.0	40386575	855550	1.39		344642
B4	793760	0.34	82663170	1307820	1839900	0.84	0.96	1.0	82663170	1307820	1.43		793760
B5	760224	0.34	82663170	1307820	1660900	0.89	1.01	1.0	82663170	1307820	1.44		760224
B6	730945	0.34	82663170	1307820	1523000	0.93	1.05	1.0	82663170	1307820	1.45		730945
B7	1620582	0.34	192918360	2185380	5707400	0.62	0.76	1.0	192918360	2185380	1.31		1620582
B8	1583023	0.34	192918360	2185380	5201300	0.65	0.79	1.0	192918360	2185380	1.33		1583023
B9	1548404	0.34	192918360	2185380	4793100	0.68	0.81	1.0	192918360	2185380	1.35		1548404
B10	2648310	0.34	341841936	3277360	13435000	0.49	0.67	1.0	341841936	3277360	1.21		2648310
B11	2609889	0.34	341841936	3277360	12391000	0.51	0.69	1.0	341841936	3277360	1.23		2609889
B12	2573134	0.34	341841936	3277360	11493000	0.53	0.70	1.0	341841936	3277360	1.24		2573134

Current EC1993-1-1 with LBA					
variable for solve function	F_{Ed}	$F_{Ed}^*h^3/2^*c_m^*bf$	$F/N_{Rd}+(n/n-1)*F_{Ed}^*e^*/M_{Rd}$	SOLVE FOR = 1, F_{Ed}	
ϕF	c_m	bound. Fact.	M_{Ed}		
N	[-]	[-]	Nmm	[-]	
B01	9306.7	0.9	5.57	39922190	1.00
B02	9005.1	0.9	0.60	40506779	1.00
B03	8734.0	0.9	0.62	40901729	1.00
B1	1589.0	0.9	0.57	4087625	1.00
B2	1489.2	0.9	0.60	4020078	1.00
B3	1407.0	0.9	0.62	3955629	1.00
B4	3240.5	0.9	0.57	8338353	1.00
B5	3103.6	0.9	0.60	8377274	1.00
B6	2984.1	0.9	0.62	8386784	1.00
B7	6616.0	0.9	0.57	17034811	1.00
B8	6462.7	0.9	0.60	17439238	1.00
B9	6321.3	0.9	0.62	1754476	1.00
B10	10811.7	0.9	0.57	27862137	1.00
B11	10654.8	0.9	0.60	28740758	1.00
B12	10504.8	0.9	0.62	29477447	1.00

Current EC1993-1-1 with Nomogram													
variable for solve function	$W_i^*f_{yd}$	A^*f_{yd}		$(E_L/L_c)/(E_L/L_b)$	$C_A^*C_m^*\lambda^{\lambda^2}\sin(\lambda)$	$(C_A+C_m)\lambda^{\lambda^2}\cos(\lambda)+\sin(\lambda)$	eq1-eq2=0						
F_{Ed}	α	M_{Rk}	N_{Rk}	μ	C_A	C_B	λ	eq1	eq2	SOLVE FOR = 0, λ			
N	[-]	Nmm	N	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]	[-]
B01	2792657	0.34	423174646	3772585	6	0	0.167	2.72	0	1.43047E-09	0.00		
B02	2720617	0.34	423174646	3772585	6	0	0.250	2.57	0	1.55102E-10	0.00		
B03	2657147	0.34	423174646	3772585	6	0	0.333	2.46	0	-7.36087E-11	0.00		
B1	465503	0.34	40386575	855550	6	0	0.167	2.72	0	1.43047E-09	0.00		
B2	441378	0.34	40386575	855550	6	0	0.250	2.57	0	1.55102E-10	0.00		
B3	421063	0.34	40386575	855550	6	0	0.333	2.46	0	-7.36086E-11	0.00		
B4	890810	0.34	82663170	1307820	6	0	0.167	2.72	0	1.43047E-09	0.00		
B5	861220	0.34	82663170	1307820	6	0	0.250	2.57	0	1.55102E-10	0.00		
B6	835388	0.34	82663170	1307820	6	0	0.333	2.46	0	-7.36086E-11	0.00		
B7	1737822	0.34	192918360	2185380	6	0	0.167	2.72	0	1.43047E-09	0.00		
B8	1704263	0.34	192918360	2185380	6	0	0.250	2.57	0	1.55102E-10	0.00		
B9	1674586	0.34	192918360	2185380	6	0	0.333	2.46	0	-7.36086E-11	0.00		
B10	2790069	0.34	341841936	3277360	6	0	0.167	2.72	0	1.43047E-09	0.00		
B11	2752452	0.34	341841936	3277360	6	0	0.250	2.57	0	1.55102E-10	0.00		
B12	2719155	0.34	341841936	3277360	6	0	0.333	2.46	0	-7.36086E-11	0.00		

Current EC1993-1-1 with Nomogram															
$\pi/\lambda (K_0)$	π/λ^*h	π^*E_L/L_{cr}	$\lambda = \sqrt{(F_{Ed}/F_{cr})}$	$\Phi = 0.5(1+\alpha^*(\lambda-0.2))+\lambda^2$	$\chi = 1/(\Phi+\sqrt{(\Phi^2-\lambda^2)})$	M_{Rk}/Y_{M0}	N_{Rk}/Y_{M0}	$1/(1-(F_{Ed}/F_{Rd})\chi^{\lambda^2})$	boundary factor	$F_{Ed}^*h^3/2^*c_m^*bf$	$F/N_{Rd}+(n/n-1)*F_{Ed}^*e^*/M_{Rd}$	SOLVE FOR = 1, F_{Ed}			
nomogram	m_{cr}	N_{cr}	λ	Φ	χ	Y_{M0}	M_{Rd}	N_{Rd}	k	ϕF	c_m	bf	M_{Ed}		
[-]	mm	N	[-]	[-]	[-]	[-]	Nmm	N	[-]	N	[-]	N	N	[-]	
B01	1	5783	10713839	0.59	0.74	0.84	1.0	423174646	3772585	1.28	10197.3	0.9	0.57	43742617	1.00
B02	1	6111	9592917	0.63	0.77	0.82	1.0	423174646	3772585	1.30	9934.3	0.9	0.60	44686523	1.00
B03	1	6397	8755266	0.66	0.79	0.81	1.0	423174646	3772585	1.32	9702.5	0.9	0.62	45437329	1.00
B1	1	3470	997902	0.93	1.05	0.64	1.0	40386575	855550	1.43	1900.4	0.9	0.57	4888586	1.00
B2	1	3667	893498	0.98	1.11	0.61	1.0	40386575	855550	1.43	1801.9	0.9	0.60	4864384	1.00
B3	1	3838	815479	1.02	1.16	0.58	1.0	40386575	855550	1.43	1719.0	0.9	0.62	4832753	1.00
B4	1	3470	2745547	0.69	0.82	0.79	1.0	82663170	1307820	1.34	3636.7	0.9	0.57	9357856	1.00
B5	1	3667	2458297	0.73	0.86	0.77	1.0	82663170	1307820	1.37	3515.9	0.9	0.60	9490190	1.00
B6	1	3838	2243640	0.76	0.89	0.75	1.0	82663170	1307820	1.39	3410.5	0.9	0.62	9585154	1.00
B7	1	3470	8926181	0.49	0.67	0.89	1.0	192918360	2185380	1.21	7094.6	0.9	0.57	18267180	1.00
B8	1	3667	7992290	0.52	0.69	0.87	1.0	192918360	2185380	1.23	6957.6	0.9	0.60	18774872	1.00
B9	1	3838	7294405	0.55	0.71	0.86	1.0	192918360	2185380	1.25	6836.5	0.9	0.62	19201313	1.00
B10	1	3470	22382783	0.38	0.60	0.93	1.0	341841936	3277360	1.13	11390.4	0.9	0.57	29353536	1.00
B11	1	3667	20041011	0.40	0.62	0.92	1.0	341841936	3277360	1.15	11236.8	0.9	0.60	30310702	1.00
B12	1	3838	18291036	0.42	0.63	0.92	1.0	341841936	3277360	1.16	11100.9	0.9	0.62	31150241	1.00

Current EC1993-1-1 with Yura													
variable for solve function	$W_i^*f_{yd}$	A^*f_{yd}	$K_0^*\sqrt{(\Sigma F+\Sigma Q)/\Sigma F}$	K^*h	π^*E_L/L_{cr}	$\lambda = \sqrt{(F_{Ed}/F_{cr})}$	$\Phi = 0.5(1+\alpha^*(\lambda-0.2))+\lambda^2$	$\chi = 1/(\Phi+\sqrt{(\Phi^2-\lambda^2)})$	M_{Rk}/Y_{M0}	N_{Rk}/Y_{M0}			
F_{Ed}	α	M_{Rk}	N_{Rk}	K	L_{cr}	N_{cr}	λ	Φ	χ	Y_{M0}	M_{Rd}	N_{Rd}	
N	[-]	Nmm	N	[-]	mm	N	[-]	[-]	[-]	[-]	Nmm	N	
B01	2557988	0.34	423174646	3772585	1.42	7082	7142559	0.73	0.85	1.0	423174646	3772585	
B02	2467319	0.34	423174646	3772585	1.50	7484	6395278	0.77	0.89	1.0	423174646	3772585	
B03	2388101	0.34	423174646	3772585	1.57	7834	5836844	0.80	0.93	1.0	423174646	3772585	
B1	384324	0.34	40386575	855550	1.42	4249	665268	1.13	1.30	1.0	40386575	855550	
B2	358989	0.34	40386575	855550	1.50	4491	595665	1.20	1.39	1.0	40386575	855550	
B3	338333	0.34	40386575	855550	1.57	4701	543652	1.25	1.47	1.0	40386575	855550	
B4	792324	0.34	82663170	1307820	1.42	4249	1830365	0.85	0.97	1.0	82663170	1307820	
B5	756346	0.34	82663170	1307820	1.50	4491	1638865	0.89	1.02	1.0	82663170	1307820	
B6	725518	0.34	82663170	1307820	1.57	4701	1495760	0.94	1.06	1.0	82663170	1307820	
B7	1633349	0.34	192918360	2185380	1.42	4249	5950787	0.61	0.75	1.0	192918360	2185380	
B8	1590961	0.34	192918360	2185380	1.50	4491	5328193	0.64	0.78	1.0	192918360	2185380	
B9	1553473	0.34	192918360	2185380	1.57	4701	4862937	0.67	0.80	1.0	192918360	2185380	
B10	2681907	0.34	341841936	3277360	1.42	4249	14921855	0.47	0.66	1.0	341841936	3277360	
B11	2635722	0.34	341841936	3277360	1.50	4491	13360674	0.50	0.67	1.0	341841936	3277360	
B12	2594702	0.34	341841936	3277360	1.57	4701	12194024	0.52	0.69	1.0	341841936	3277360	

Current EC1993-1-1 with Yura					
$1/(1-(F_{Ed}/F_{Rd})\chi^{\lambda^2})$	boundary factor	$F_{Ed}^*h^3/2^*c_m^*bf$			

Current EC1993-1-1 with L.F. Geschwindner														
K ³ h	$\pi^2 E I / L_{cr}^2$	$\lambda = \sqrt{(F_{Rd} / F_{cr})}$	$\Phi = 0.5(1 + \alpha^*(\lambda - 0.2) + \lambda^2)$	$\chi = 1 / (\Phi + \sqrt{\Phi^2 - \lambda^2})$		M_{Rd} / Y_{M0}	N_{Rd} / Y_{M0}	$1 / (1 - (F_{ult} / F_{Rd}) * \chi^2 \lambda^2)$			boundary factor	$F_{ult} * h^3 / 2^2 * C_m * bf$	$F / N_{Rd} + (n / (n-1)) * F_{ult} * e' / M_{Rd}$	
L_{cr}	N_{cr}	λ	Φ	χ	Y_{M0}	M_{Rd}	N_{Rd}	k	φF	c_m	bf	M_{Rd}	SOLVE FOR = 1, F_{ult}	
mm	N	[-]	[-]	[-]	[-]	Nmm	N	[-]	N	[-]	[-]	Nmm	[-]	
801	6466	8568908	0.66	0.80	0.80	1.0	423174646	3772585	1.33	9758.4	0.9	0.57	41859676	1.00
802	6707	7964255	0.69	0.82	0.79	1.0	423174646	3772585	1.35	9544.6	0.9	0.60	42933701	1.00
803	6930	7459693	0.71	0.84	0.78	1.0	423174646	3772585	1.36	9348.8	0.9	0.62	43780879	1.00
81	3880	798120	1.04	1.18	0.57	1.0	40386575	855550	1.44	1722.7	0.9	0.57	4431478	1.00
82	4024	741802	1.07	1.23	0.55	1.0	40386575	855550	1.43	1651.1	0.9	0.60	4457319	1.00
83	4158	694806	1.11	1.27	0.53	1.0	40386575	855550	1.42	1587.9	0.9	0.62	4464104	1.00
84	3880	2195883	0.77	0.89	0.74	1.0	82663170	1307820	1.40	3429.0	0.9	0.57	8823463	1.00
85	4024	2040934	0.80	0.92	0.72	1.0	82663170	1307820	1.41	3333.2	0.9	0.60	8996890	1.00
86	4158	1911634	0.83	0.95	0.71	1.0	82663170	1307820	1.42	3246.1	0.9	0.62	9123195	1.00
87	3880	7139142	0.55	0.71	0.86	1.0	192918360	2185380	1.25	6876.3	0.9	0.57	17705118	1.00
88	4024	6635379	0.57	0.73	0.85	1.0	192918360	2185380	1.27	6763.5	0.9	0.60	18250937	1.00
89	4158	6215006	0.59	0.74	0.84	1.0	192918360	2185380	1.28	6659.7	0.9	0.62	18704773	1.00
810	3880	17901706	0.43	0.63	0.91	1.0	341841936	3277360	1.16	11162.9	0.9	0.57	28767318	1.00
811	4024	16638499	0.44	0.64	0.91	1.0	341841936	3277360	1.17	11035.5	0.9	0.60	29767570	1.00
812	4158	15584394	0.46	0.65	0.90	1.0	341841936	3277360	1.18	10918.2	0.9	0.62	30637541	1.00

APPENDIX D – SCRIPT

The scripts below are the parametric scripts for all frame configurations with a leaning column, frames A, A-V, A-H, A-VH, and B. The input is generated from an Excel sheet, as given in Appendix B. For each analysis type (GMNIA IV, GMNIA III, and LBA), a different script is used.

```

1  # -*- coding: mbc8 -*-
2  from part import *
3  from material import *
4  from section import *
5  from assembly import *
6  from step import *
7  from interaction import *
8  from load import *
9  from mesh import *
10 from optimization import *
11 from job import *
12 from sketch import *
13 from visualization import *
14 from connectorBehavior import *
15 from caeModules import *
16 import os
17 import sys
18 import csv
19 import odbAccess
20 import math
21 import numpy as np
22 import xlswriter
23
24 session.journalOptions.setValue(replayGeometry=COORDINATE, recoverGeometry=COORDINATE)
25 # =====setting up=====
26
27 input_file=open('C:/Users/20174572/Google Drive/Wim Broeks/TUe/2. Master/Afstuderen/4. FE Modelling/4.
Parametric/INPUT.txt')
28
29 for line in input_file:
30     extracted_line = line
31     extracted_list = extracted_line.split()
32     if "id" not in extracted_line:
33         id = extracted_list[0]
34         E = float(extracted_list[1])
35         pois = float(extracted_list[2])
36         F = float(extracted_list[3])
37         Q = float(extracted_list[4])
38         H = float(extracted_list[5])
39         fy = float(extracted_list[6])
40         fu = float(extracted_list[7])
41         strain = float(extracted_list[8])
42         h = float(extracted_list[9])
43         b1 = float(extracted_list[10])
44         b2 = float(extracted_list[11])
45         P1h = float(extracted_list[16])
46         P1b = float(extracted_list[17])
47         P1tw = float(extracted_list[18])
48         P1tf = float(extracted_list[19])
49         P2h = float(extracted_list[27])
50         P2b = float(extracted_list[28])
51         P2tw = float(extracted_list[29])
52         P2tf = float(extracted_list[30])
53         P4h = float(extracted_list[38])
54         P4b = float(extracted_list[39])
55         P4tw = float(extracted_list[40])
56         P4tf = float(extracted_list[41])
57         P5h = float(extracted_list[49])
58         P5b = float(extracted_list[50])
59         P5tw = float(extracted_list[51])
60         P5tf = float(extracted_list[52])
61         joint = float(extracted_list[59])
62         condition = float(extracted_list[60])
63         ma1 = float(extracted_list[62])
64         mb1 = float(extracted_list[63])
65         mc1 = float(extracted_list[64])
66         rs1 = float(extracted_list[65])
67         ma2 = float(extracted_list[66])
68         mb2 = float(extracted_list[67])
69         mc2 = float(extracted_list[68])
70         rs2 = float(extracted_list[69])
71         ma4 = float(extracted_list[70])
72         mb4 = float(extracted_list[71])
73         mc4 = float(extracted_list[72])
74         rs4 = float(extracted_list[73])
75         ma5 = float(extracted_list[74])
76         mb5 = float(extracted_list[75])
77         mc5 = float(extracted_list[76])
78         rs5 = float(extracted_list[77])
79
80         model = extracted_list[80]
81         mode2 = extracted_list[81]
82         mode3 = extracted_list[82]
83         A1 = float(extracted_list[126])
84         A2 = float(extracted_list[127])
85         A3 = float(extracted_list[128])

```

```

86
87     #mesh
88     P1sy = ma1
89     P1sx = mb1
90     P1sz = mc1
91
92     P2sy = mb2
93     P2sx = ma2
94     P2sz = mc2
95
96     P3sx = mb1
97     P3sz = mc1
98
99     P4sy = ma4
100    P4sx = mb4
101    P4sz = mc4
102
103    P5sy = mb5
104    P5sx = ma5
105    P5sz = mc5
106
107    P6sx = mb4
108    P6sz = mc4
109
110    # =====setting up=====
111    Filename='Frame-'+id
112    Loadpath='F:\\Graduation project Wim\\parametric study\\plot\\{0}'.format(Filename)
113    if not os.path.exists(Loadpath):
114        os.makedirs(Loadpath)
115    os.chdir(Loadpath)
116
117    Mymodel=mdb.Model(name='GMNIA_IV-frame-'+id)
118    Myassembly=Mymodel.rootAssembly
119
120    #part 1 - stabilizing columns
121    Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
122    Mysketchs=Mymodel.sketches['__sweep__']
123    Mysketchs.Line(point1=(0.0, 0.0), point2=(0.0, h+P2h/2))
124    Mysketchs.geometry.findAt((0.0, h/2))
125    Mysketchs.VerticalConstraint(addUndoState=False, entity=Mysketchs.geometry.findAt((0.0, h/2), ))
126    Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(1.0, 0.0, 0.0, 0.0, 0.0,
1.0, -0.0, -1.0, -0.0, 0.0, 0.0, 0.0))
127    Mysketchp=Mymodel.sketches['__profile__']
128    Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
129    Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
130    Mysketchp.Line(point1=(-(P1h/2), (P1b/2)), point2=(-(P1h/2), -(P1b/2)))
131    Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[4])
132    Mysketchp.Line(point1=((P1h/2), (P1b/2)), point2=((P1h/2), -(P1b/2)))
133    Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[5])
134    Mysketchp.Line(point1=(-(P1h/2), 0.0), point2=((P1h/2), 0.0))
135    Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry[6])
136    Mymodel.Part(dimensionality=THREE_D, name='Part-1', type=DEFORMABLE_BODY)
137    Mypart1=Mymodel.parts['Part-1']
138    Mypart1.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
139
140    Mypart1.DatumPlaneByPrincipalPlane(offset=(h-(P2h/2)), principalPlane=XZPLANE)
141    Mypart1.DatumPlaneByPrincipalPlane(offset=(h+(P2h/2)), principalPlane=XZPLANE)
142    Mypart1.DatumPlaneByPrincipalPlane(offset=(h), principalPlane=XZPLANE)
143
144    #part 2 - stabilizing beam
145    Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
146    Mysketchs=Mymodel.sketches['__sweep__']
147    Mysketchs.Line(point1=(P1h/2, h), point2=((b1-P1h/2), h))
148    Mysketchs.HorizontalConstraint(addUndoState=False, entity=Mysketchs.geometry[2])
149    Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(0.0, -1.0, 0.0, -0.0, 0.0,
1.0, -1.0, -0.0, -0.0, 0, h, 0.0))
150    Mysketchp=Mymodel.sketches['__profile__']
151    Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
152    Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
153    Mysketchp.Line(point1=(-(P2h/2), (P2b/2)), point2=(-(P2h/2), -(P2b/2)))
154    Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[4])
155    Mysketchp.Line(point1=((P2h/2), (P2b/2)), point2=((P2h/2), -(P2b/2)))
156    Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[5])
157    Mysketchp.Line(point1=(-(P2h/2), 0.0), point2=((P2h/2), 0.0))
158    Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry[6])
159    Mymodel.Part(dimensionality=THREE_D, name='Part-2', type=DEFORMABLE_BODY)
160    Mypart2=Mymodel.parts['Part-2']
161    Mypart2.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
162
163    #part 3a - stiffeners stabilizing column
164    Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
165    Mysketchp=Mymodel.sketches['__profile__']
166    Mysketchp.Line(point1=(0.0, 0.0), point2=(P1h, 0.0))
167    Mysketchp.geometry.findAt((P1h/2, 0.0))
168    Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P1h/2, 0.0), ))
169    Mymodel.Part(dimensionality=THREE_D, name='Part-3a', type=DEFORMABLE_BODY)

```

```

170 Mypart3a=Mymodel.parts['Part-3a']
171 Mypart3a.BaseShellExtrude(depth=P1b, sketch=Mysketchp)
172 Mypart3a.DatumPlaneByPrincipalPlane(offset=(P1b/2), principalPlane=XYPLANE)
173
174 #part 3b - stiffeners stabilizing column
175 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
176 Mysketchp=Mymodel.sketches['__profile__']
177 Mysketchp.Line(point1=(0.0, 0.0), point2=(P1h, 0.0))
178 Mysketchp.geometry.findAt((P1h/2, 0.0))
179 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P1h/2, 0.0), ))
180 Mymodel.Part(dimensionality=THREE_D, name='Part-3b', type=DEFORMABLE_BODY)
181 Mypart3b=Mymodel.parts['Part-3b']
182 Mypart3b.BaseShellExtrude(depth=P1b, sketch=Mysketchp)
183 Mypart3b.DatumPlaneByPrincipalPlane(offset=(P1b/2), principalPlane=XYPLANE)
184
185 #part 4 - leaning column
186 Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
187 Mysketchs=Mymodel.sketches['__sweep__']
188 Mysketchs.Line(point1=(b1+b2, 0.0), point2=(b1+b2, h+P5h/2))
189 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(1.0, 0.0, 0.0, 0.0, 0.0,
1.0, -0.0, -1.0, -0.0, b1+b2, 0.0, 0.0))
190 Mysketchp=Mymodel.sketches['__profile__']
191 Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
192 Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
193 Mysketchp.Line(point1=(-P4h/2, P4b/2), point2=(-P4h/2, -P4b/2))
194 Mysketchp.Line(point1=(P4h/2, P4b/2), point2=(P4h/2, -P4b/2))
195 Mysketchp.Line(point1=(-P4h/2, 0.0), point2=(P4h/2, 0.0))
196 Mymodel.Part(dimensionality=THREE_D, name='Part-4', type=DEFORMABLE_BODY)
197 Mypart4=Mymodel.parts['Part-4']
198 Mypart4.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
199
200 Mypart4.DatumPlaneByPrincipalPlane(offset=h-P5h/2, principalPlane=XZPLANE)
201 Mypart4.DatumPlaneByPrincipalPlane(offset=h+P5h/2, principalPlane=XZPLANE)
202 Mypart4.DatumPlaneByPrincipalPlane(offset=(h), principalPlane=XZPLANE)
203
204 #part 5 - leaning beam
205 Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
206 Mysketchs=Mymodel.sketches['__sweep__']
207 Mysketchs.Line(point1=(b1+P1h/2, h), point2=(b1+b2-P4h/2, h))
208 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(0.0, -1.0, 0.0, -0.0, 0.0,
1.0, -1.0, -0.0, -0.0, b1, h, 0.0))
209 Mysketchp=Mymodel.sketches['__profile__']
210 Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
211 Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
212 Mysketchp.Line(point1=(-P5h/2, P5b/2), point2=(-P5h/2, -P5b/2))
213 Mysketchp.Line(point1=(P5h/2, P5b/2), point2=(P5h/2, -P5b/2))
214 Mysketchp.Line(point1=(-P5h/2, 0.0), point2=(P5h/2, 0.0))
215 Mymodel.Part(dimensionality=THREE_D, name='Part-5', type=DEFORMABLE_BODY)
216 Mypart5=Mymodel.parts['Part-5']
217 Mypart5.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
218
219 if (joint ==1):
220 #part 6a - stiffeners leaning column
221 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
222 Mysketchp=Mymodel.sketches['__profile__']
223 Mysketchp.Line(point1=(0.0, 0.0), point2=(P4h, 0.0))
224 Mysketchp.geometry.findAt((P4h/2, 0.0))
225 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P4h/2, 0.0),
))
Mymodel.Part(dimensionality=THREE_D, name='Part-6a', type=DEFORMABLE_BODY)
227 Mypart6a=Mymodel.parts['Part-6a']
228 Mypart6a.BaseShellExtrude(depth=P4b, sketch=Mysketchp)
229 Mypart6a.DatumPlaneByPrincipalPlane(offset=(P4b/2), principalPlane=XYPLANE)
230
231 elif (joint ==2):
232 #part 6a - stiffeners leaning column
233 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
234 Mysketchp=Mymodel.sketches['__profile__']
235 Mysketchp.Line(point1=(0.0, 0.0), point2=(P4h, 0.0))
236 Mysketchp.geometry.findAt((P4h/2, 0.0))
237 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P4h/2, 0.0),
))
Mymodel.Part(dimensionality=THREE_D, name='Part-6a', type=DEFORMABLE_BODY)
239 Mypart6a=Mymodel.parts['Part-6a']
240 Mypart6a.BaseShellExtrude(depth=P4b, sketch=Mysketchp)
241 Mypart6a.DatumPlaneByPrincipalPlane(offset=(P4b/2), principalPlane=XYPLANE)
242 #part 6b - stiffeners leaning column
243 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
244 Mysketchp=Mymodel.sketches['__profile__']
245 Mysketchp.Line(point1=(0.0, 0.0), point2=(P4h, 0.0))
246 Mysketchp.geometry.findAt((P4h/2, 0.0))
247 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P4h/2, 0.0),
))
Mymodel.Part(dimensionality=THREE_D, name='Part-6b', type=DEFORMABLE_BODY)
249 Mypart6b=Mymodel.parts['Part-6b']
250 Mypart6b.BaseShellExtrude(depth=P4b, sketch=Mysketchp)

```

```

251         Mypart6b.DatumPlaneByPrincipalPlane (offset=(P4b/2), principalPlane=XYPLANE)
252
253     #assembly
254     Myassembly.DatumCsysByDefault (CARTESIAN)
255     Myassembly.Instance (dependent=ON, name='Part-1-1', part=Mypart1)
256     Myassembly.Instance (dependent=ON, name='Part-2-1', part=Mypart2)
257     Myassembly.Instance (dependent=ON, name='Part-1-2', part=Mypart1)
258     Myassembly.translate (instanceList=('Part-1-2', ), vector=(b1, 0.0, 0.0))
259     Myassembly.Instance (dependent=ON, name='Part-4-1', part=Mypart4)
260     Myassembly.Instance (dependent=ON, name='Part-5-1', part=Mypart5)
261
262     #partition
263     #part 1
264     Mypart1.PartitionFaceByDatumPlane (datumPlane=Mypart1.datums[2], faces=Mypart1.faces.findAt (((-P1h/2,
h/4, P1b/4), ), ((-P1h/2, h/4, -P1b/4), ), ((P1h/2, h/4, P1b/4), ), ((P1h/2, h/4, -P1b/4), ), ((P1h/4
, h/4, 0.0), ), ))
265     #part 4
266     Mypart4.PartitionFaceByDatumPlane (datumPlane=Mypart4.datums[2], faces=Mypart4.faces.findAt (((b1+b2-
P4h/2, h/4, P4b/4), ), ((b1+b2-P4h/2, h/4, -P4b/4), ), ((b1+b2+P4h/2, h/4, P4b/4), ), ((b1+b2+P4h/2,
h/4, -P4b/4), ), ((b1+b2, h/4, 0.0), ), ))
267
268     #part 3a-1
269     Mypart3a.PartitionFaceByDatumPlane (datumPlane=Mypart3a.datums[2], faces=Mypart3a.faces.findAt (((P1h/4
, 0.0, P1b/4), ))
270     Myassembly.Instance (dependent=ON, name='Part-3a-1', part=Mypart3a)
271     Mymodel.Tie (adjust=ON, master=Region (edges=Myassembly.instances['Part-1-1'].edges.findAt (((-P1h/2, h
+(P2h/2), P1b/4), ), ((P1h/2, h+(P2h/2), P1b/4), ), ((-P1h/2, h+(P2h/2), -P1b/4), ), ((P1h/2, h
+(P2h/2), -P1b/4), ), ((0.0, h+(P2h/2), 0.0), ), )), name='Tie_A_a', positionToleranceMethod=COMPUTED,
slave=Region (edges=Myassembly.instances['Part-3a-1'].edges.findAt (((0.0, 0.0, P1b/4), ), ((P1h, 0.0,
P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h/2, 0.0, P1b/2), ), )), thickness=
ON, tieRotations=ON)
272     Myassembly.translate (instanceList=('Part-3a-1', ), vector=(-P1h/2, h+(P2h/2), -(P1b/2)))
273
274     #part 3b-1
275     Mypart3b.PartitionFaceByDatumPlane (datumPlane=Mypart3b.datums[2], faces=Mypart3b.faces.findAt (((P1h/4
, 0.0, P1b/4), ))
276     Myassembly.Instance (dependent=ON, name='Part-3b-1', part=Mypart3b)
277     Mymodel.Tie (adjust=ON, master=Region (edges=Myassembly.instances['Part-1-1'].edges.findAt (((-P1h/2),
h-(P2h/2), P1b/4), ), ((P1h/2, h-(P2h/2), P1b/4), ), ((-P1h/2, h-(P2h/2), -P1b/4), ), ((P1h/2,
h-(P2h/2), -P1b/4), ), ((0.0, h-(P2h/2), 0.0), ), )), name='Tie_A_b', positionToleranceMethod=
COMPUTED, slave=Region (edges=Myassembly.instances['Part-3b-1'].edges.findAt (((0.0, 0.0, P1b/4), ), ((
P1h, 0.0, P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h/2, 0.0, P1b/2), ), )),
thickness=ON, tieRotations=ON)
278     Myassembly.translate (instanceList=('Part-3b-1', ), vector=(-P1h/2, (h-(P2h/2)), -(P1b/2)))
279
280     #part 3a-2
281     Myassembly.Instance (dependent=ON, name='Part-3a-2', part=Mypart3a)
282     Mymodel.Tie (adjust=ON, master=Region (edges=Myassembly.instances['Part-1-2'].edges.findAt (((-P1h/2)+
b1, h+(P2h/2), P1b/4), ), ((P1h/2)+b1, h+(P2h/2), P1b/4), ), ((-P1h/2)+b1, h+(P2h/2), -P1b/4), ),
(((P1h/2)+b1, h+(P2h/2), -P1b/4), ), ((b1, h+(P2h/2), 0.0), ), )), name='Tie_B_a',
positionToleranceMethod=COMPUTED, slave=Region (edges=Myassembly.instances['Part-3a-2'].edges.findAt
(((0.0, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h
/2, 0.0, P1b/2), ), )), thickness=ON, tieRotations=ON)
283     Myassembly.translate (instanceList=('Part-3a-2', ), vector=(-P1h/2+b1, (h+(P2h/2)), -(P1b/2)))
284
285     #part 3b-2
286     Myassembly.Instance (dependent=ON, name='Part-3b-2', part=Mypart3b)
287     Mymodel.Tie (adjust=ON, master=Region (edges=Myassembly.instances['Part-1-2'].edges.findAt (((-P1h/2)+
b1, h-(P2h/2), P1b/4), ), ((P1h/2)+b1, h-(P2h/2), P1b/4), ), ((-P1h/2)+b1, h-(P2h/2), -P1b/4), ),
(((P1h/2)+b1, h-(P2h/2), -P1b/4), ), ((b1, h-(P2h/2), 0.0), ), )), name='Tie_B_b',
positionToleranceMethod=COMPUTED, slave=Region (edges=Myassembly.instances['Part-3b-2'].edges.findAt
(((0.0, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h
/2, 0.0, P1b/2), ), )), thickness=ON, tieRotations=ON)
288     Myassembly.translate (instanceList=('Part-3b-2', ), vector=(-P1h/2+b1, (h-(P2h/2)), -(P1b/2)))
289
290     if (joint ==1):
291         #part 6-1
292         Mypart6a.PartitionFaceByDatumPlane (datumPlane=Mypart6a.datums[2], faces=Mypart6a.faces.findAt (((
P4h/4, 0.0, P4b/4), ))
293         Myassembly.Instance (dependent=ON, name='Part-6a-1', part=Mypart6a)
294         Mymodel.Tie (adjust=ON, master=Region (edges=Myassembly.instances['Part-4-1'].edges.findAt (((-P4h/
2)+b1+b2, h+(P5h/2), P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), P4b/4), ), ((-P4h/2)+b1+b2, h+(P5h/2
), -P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), -P4b/4), ), ((b1+b2, h+(P5h/2), 0.0), ), )), name=
'Tie_C_a', positionToleranceMethod=COMPUTED, slave=Region (edges=Myassembly.instances['Part-6a-1'
].edges.findAt (((0.0, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0,
P4b/4*3), ), ((P4h/2, 0.0, P4b/2), ), )), thickness=ON, tieRotations=ON)
295         Myassembly.translate (instanceList=('Part-6a-1', ), vector=(-P4h/2+b1+b2, (h+(P5h/2)), -(P4b/2)))
296
297     elif (joint ==2):
298         #part 6-1
299         Mypart6a.PartitionFaceByDatumPlane (datumPlane=Mypart6a.datums[2], faces=Mypart6a.faces.findAt (((
P4h/4, 0.0, P4b/4), ))
300         Myassembly.Instance (dependent=ON, name='Part-6a-1', part=Mypart6a)
301         Mymodel.Tie (adjust=ON, master=Region (edges=Myassembly.instances['Part-4-1'].edges.findAt (((-P4h/
2)+b1+b2, h+(P5h/2), P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), P4b/4), ), ((-P4h/2)+b1+b2, h+(P5h/2
), -P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), -P4b/4), ), ((b1+b2, h+(P5h/2), 0.0), ), )), name=

```



```

    'Tie_C a', positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-6a-1']
    ].edges.findAt(((0.0, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0,
    P4b/4*3), ), ((P4h/2, 0.0, P4b/2), ), )), thickness=ON, tieRotations=ON)
301 Myassembly.translate(instanceList=('Part-6a-1', ), vector=(-P4h/2+b1+b2, (h+(P5h/2)), -(P4b/2)))
302 #part 6-2
303 Mypart6b.PartitionFaceByDatumPlane(datumPlane=Mypart6b.datums[2], faces=Mypart6b.faces.findAt(((
    P4h/4, 0.0, P4b/4), )))
304 Myassembly.Instance(dependent=ON, name='Part-6b-1', part=Mypart6b)
305 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-4-1'].edges.findAt(((-(P4h/
    2)+b1+b2, h-(P5h/2), P4b/4), ), ((P4h/2)+b1+b2, h-(P5h/2), P4b/4), ), (-(P4h/2)+b1+b2, h-(P5h/2
    ), -P4b/4), ), ((P4h/2)+b1+b2, h-(P5h/2), -P4b/4), ), ((b1+b2, h-(P5h/2), 0.0), ), )), name=
    'Tie_C b', positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-6b-1']
    ].edges.findAt(((0.0, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0,
    P4b/4*3), ), ((P4h/2, 0.0, P4b/2), ), )), thickness=ON, tieRotations=ON)
306 Myassembly.translate(instanceList=('Part-6b-1', ), vector=(-P4h/2+b1+b2, (h-(P5h/2)), -(P4b/2)))

307
308
309 #interaction
310 #reference points
311 Myassembly.ReferencePoint(point=(0.0, h, 0.0))
312 Myassembly.ReferencePoint(point=(b1, h, 0.0))
313 Myassembly.ReferencePoint(point=(b1+b2, h, 0.0))
314 #part 1-1 with 2-1
315 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, 1000.0, 0.0))
316 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-1-1'].faces.findAt(((P1h/2, h,
    P1b/4), ), ((P1h/2, h, -P1b/4), ), )), name='Constraint-7', positionToleranceMethod=COMPUTED, slave=
    Region(edges=Myassembly.instances['Part-2-1'].edges.findAt(((P1h/2, h+P2h/2+1000, P2b/4), ), ((P1h/2,
    h+P2h/2+1000, -P2b/4), ), ((P1h/2, h-P2h/2+1000, P2b/4), ), ((P1h/2, h-P2h/2+1000, -P2b/4), ), ((P1h
    /2, h+1000, 0.0), ), )), thickness=ON, tieRotations=ON)
317 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, -1000.0, 0.0))
318
319 #part 1-2 with 2-1
320 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, 1000.0, 0.0))
321 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-1-2'].faces.findAt(((b1-P1h/2,
    h, P1b/4), ), ((b1-P1h/2, h, -P1b/4), ), )), name='Constraint-8', positionToleranceMethod=COMPUTED,
    slave=Region(edges=Myassembly.instances['Part-2-1'].edges.findAt(((b1-P1h/2, h-P2h/2+1000, -P1b/4),
    ), ((b1-P1h/2, h-P2h/2+1000, P1b/4), ), ((b1-P1h/2, h+1000, 0.0), ), ((b1-P1h/2, h+P2h/2+1000, P1b/4
    ), ), ((b1-P1h/2, h+P2h/2+1000, -P1b/4), ), )), thickness=ON, tieRotations=ON)
322 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, -1000.0, 0.0))
323
324 #part 1-2 with 5-1
325 Mypart5.PartitionEdgeByPoint(edge=Mypart5.edges.findAt((b1+P1h/2, h, 0.0), ), point=Mypart5.
    InterestingPoint(Mypart5.edges.findAt((b1+P1h/2, h, 0.0), ), MIDDLE))
326 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, 1000.0, 0.0))
327 Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-5-1'].vertices.findAt(((b1+
    P1h/2, h+1000, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name
    ='Constraint-9', surface=Region(edges=Myassembly.instances['Part-5-1'].edges.findAt(((b1+P1h/2, h+P5h
    /2+1000, P5b/4), ), ((b1+P1h/2, h+P5h/2+1000, -P5b/4), ), ((b1+P1h/2, h-P5h/2+1000, P5b/4), ), ((b1+
    P1h/2, h-P5h/2+1000, -P5b/4), ), ((b1+P1h/2, h+P5h/4+1000, 0.0), ), ((b1+P1h/2, h-P5h/4+1000, 0.0),
    ), )), u1=ON, u2=ON, u3=ON, ur1=ON, ur2=ON, ur3=ON)
328 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-1-2'].faces.findAt(((b1+P1h/2,
    h, P1b/4), ), ((b1+P1h/2, h, -P1b/4), ), )), name='Constraint-10', positionToleranceMethod=COMPUTED,
    slave=Region(vertices=Myassembly.instances['Part-5-1'].vertices.findAt(((b1+P1h/2, h+1000, 0.0), )),
    thickness=ON, constraintRatio=10000.0, constraintRatioMethod=SPECIFIED, tieRotations=OFF)
329 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, -1000.0, 0.0))
330
331 #part 4-1 with 5-1
332 if (joint ==1):
333 Mypart5.PartitionEdgeByPoint(edge=Mypart5.edges.findAt((b1+b2-P4h/2, h, 0.0), ), point=Mypart5.
    InterestingPoint(Mypart5.edges.findAt((b1+b2-P4h/2, h, 0.0), ), MIDDLE))
334 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, 1000.0, 0.0))
335 Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-5-1'].vertices.findAt(((
    b1+b2-P4h/2, h+1000, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=
    None, name='Constraint-11', surface=Region(edges=Myassembly.instances['Part-5-1'].edges.findAt(((
    b1+b2-P4h/2, h+P5h/2+1000, P5b/4), ), ((b1+b2-P4h/2, h+P5h/2+1000, -P5b/4), ), ((b1+b2-P4h/2, h-
    P5h/2+1000, P5b/4), ), ((b1+b2-P4h/2, h-P5h/2+1000, -P5b/4), ), ((b1+b2-P4h/2, h+P5h/4+1000, 0.0
    ), ), ((b1+b2-P4h/2, h-P5h/4+1000, 0.0), ), )), u1=ON, u2=ON, u3=ON, ur1=ON, ur2=ON, ur3=ON)
336 Mymodel.Tie(adjust=ON, constraintRatio=10000.0, constraintRatioMethod=SPECIFIED, master=Region(
    faces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2-P4h/2, h, P4b/4), ), ((b1+b2-P4h/2, h
    , -P4b/4), ), )), name='Constraint-12', positionToleranceMethod=COMPUTED, slave=Region(vertices=
    Myassembly.instances['Part-5-1'].vertices.findAt(((b1+b2-P4h/2, h+1000, 0.0), )), thickness=ON,
    tieRotations=OFF)
337 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, -1000.0, 0.0))
338 elif (joint ==2):
339 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2-
    P4h/2, h, P4b/4), ), ((b1+b2-P4h/2, h, -P4b/4), ), )), name='Constraint-11',
    positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-5-1'].edges.
    findAt(((b1+b2-P4h/2, h+P5h/2, P5b/4), ), ((b1+b2-P4h/2, h+P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-
    P5h/2, P5b/4), ), ((b1+b2-P4h/2, h, 0.0), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/4), ), )), thickness=ON
    , tieRotations=ON)
340
341 #coupling reference points
342 if (joint ==1):
343 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[32], )),
    couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-16',

```

```

344     surface=Region(sidelFaces=Myassembly.instances['Part-1-1'].faces.findAt(((P1h/4, h, 0.0), (0.0,
0.0, 1.0)), ((-P1h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3=ON)
Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[33], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-17',
surface=Region(sidelFaces=Myassembly.instances['Part-1-2'].faces.findAt(((b1+P1h/4, h, 0.0), (0.0
, 0.0, 1.0))), ((b1-P1h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3
=ON)
345 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[34], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-18',
surface=Region(sidelFaces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2+P4h/4, h, 0.0), (
0.0, 0.0, 1.0))), ((b1+b2-P4h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=
ON, ur3=ON)
346 elif (joint ==2):
347     Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[36], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-16',
surface=Region(sidelFaces=Myassembly.instances['Part-1-1'].faces.findAt(((P1h/4, h, 0.0), (0.0,
0.0, 1.0))), ((-P1h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3=ON)
348 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[37], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-17',
surface=Region(sidelFaces=Myassembly.instances['Part-1-2'].faces.findAt(((b1+P1h/4, h, 0.0), (0.0
, 0.0, 1.0))), ((b1-P1h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3
=ON)
349 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[38], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-18',
surface=Region(sidelFaces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2+P4h/4, h, 0.0), (
0.0, 0.0, 1.0))), ((b1+b2-P4h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=
ON, ur3=ON)
350
351 #cross-sections
352 #part 1
353 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P1tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P1tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
354 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P1tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P1tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
355 #part 2
356 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P2tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P2tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
357 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P2tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P2tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
358 #part 4
359 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel leaning column', name='P4tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF,
temperature=GRADIENT, thickness=P4tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM
, useDensity=OFF)
360 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel leaning column', name='P4tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF,
temperature=GRADIENT, thickness=P4tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM
, useDensity=OFF)
361 #part 5
362 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P5tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P5tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
363 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P5tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P5tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
364
365 #section assignment
366 #part 1
367 #bottom flange
368 Mypart1.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
Mypart1.faces.findAt(((P1h/2, h/4, -P1b/4), ), ((P1h/2, h, P1b/4), ), ((P1h/2, h, -P1b/4), ), ((P1h/2
, h/4, P1b/4), ), )), sectionName='P1tf', thicknessAssignment=FROM_SECTION)
369 #top flange
370 Mypart1.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart1.faces.findAt(((P1h/2, h/4, -P1b/4), ), ((-P1h/2, h, P1b/4), ), ((-P1h/2, h, -P1b/4), ), ((-
P1h/2, h/4, P1b/4), ), )), sectionName='P1tf', thicknessAssignment=FROM_SECTION)
371 #web
372 Mypart1.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
Mypart1.faces.findAt(((0.0, h/4, 0.0), ), ((0, h, 0.0), ), )), sectionName='P1tw',
thicknessAssignment=FROM_SECTION)
373
374 #part 2
375 #bottom flange
376 Mypart2.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
Mypart2.faces.findAt(((b1/2, h-(P2h/2), -P2b/4), ), ((b1/2, h-(P2h/2), P2b/4), ), )), sectionName=
'P2tf', thicknessAssignment=FROM_SECTION)
377 #top flange
378 Mypart2.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart2.faces.findAt(((b1/2, h+(P2h/2), -P2b/4), ), ((b1/2, h+(P2h/2), P2b/4), ), )), sectionName=
'P2tf', thicknessAssignment=FROM_SECTION)
379 #web
380 Mypart2.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=

```

```

381 Mypart2.faces.findAt(((b1/2, h, 0.0), )), sectionName='P2tw', thicknessAssignment=FROM_SECTION)
382
383 #part 3a
384 Mypart3a.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
385 Mypart3a.faces.getByBoundingBox(0.0, 0.0, 0.0, P1h, 0.0, P1b)), sectionName='P2tf',
386 thicknessAssignment=FROM_SECTION)
387
388 #part 3b
389 Mypart3b.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces
390 =Mypart3b.faces.getByBoundingBox(0.0, 0.0, 0.0, P1h, 0.0, P1b)), sectionName='P2tf',
391 thicknessAssignment=FROM_SECTION)
392
393 #part 4
394 #bottom flange
395 Mypart4.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
396 Mypart4.faces.findAt(((b1+b2-P4h/2, h/4, -P4b/4), ), ((b1+b2-P4h/2, h, P4b/4), ), ((b1+b2-P4h/2, h/4,
397 P4b/4), ), ((b1+b2-P4h/2, h, -P4b/4), ), )), sectionName='P4tf', thicknessAssignment=FROM_SECTION)
398
399 #top flange
400 Mypart4.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
401 Mypart4.faces.findAt(((b1+b2+P4h/2, h/4, -P4b/4), ), ((b1+b2+P4h/2, h/4, P4b/4), ), ((b1+b2+P4h/2, h,
402 P4b/4), ), ((b1+b2+P4h/2, h, -P4b/4), ), )), sectionName='P4tf', thicknessAssignment=FROM_SECTION)
403
404 #web
405 Mypart4.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
406 Mypart4.faces.findAt(((b1+b2, h, 0.0), ), ((b1+b2, h/4, 0.0), ), )), sectionName='P4tw',
407 thicknessAssignment=FROM_SECTION)
408
409 #part 5
410 #bottom flange
411 Mypart5.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
412 Mypart5.faces.findAt(((b1+b2/2, h-P5h/2, P5b/4), ), ((b1+b2/2, h-P5h/2, -P5b/4), ), )), sectionName=
413 'P5tf', thicknessAssignment=FROM_SECTION)
414
415 #top flange
416 Mypart5.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
417 Mypart5.faces.findAt(((b1+b2/2, h+P5h/2, P5b/4), ), ((b1+b2/2, h+P5h/2, -P5b/4), ), )), sectionName=
418 'P5tf', thicknessAssignment=FROM_SECTION)
419
420 #web
421 Mypart5.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
422 Mypart5.faces.findAt(((b1+b2/2, h, 0.0), )), sectionName='P5tw', thicknessAssignment=FROM_SECTION)
423
424 if (joint==1):
425 #part 6a
426 Mypart6a.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(
427 faces=Mypart6a.faces.getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b)), sectionName='P5tf',
428 thicknessAssignment=FROM_SECTION)
429
430 elif (joint ==2):
431 #part 6a
432 Mypart6a.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(
433 faces=Mypart6a.faces.getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b)), sectionName='P5tf',
434 thicknessAssignment=FROM_SECTION)
435
436 #part 6b
437 Mypart6b.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(
438 faces=Mypart6b.faces.getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b)), sectionName='P5tf',
439 thicknessAssignment=FROM_SECTION)
440
441 #material
442 Mymodel.Material(name='Steel')
443 Mymodel.materials['Steel'].Elastic(table=((E, pois), ))
444 Mymodel.materials['Steel'].Plastic(table=((fy, 0.0), (fu, strain)))
445 Mymodel.Material(name='Steel leaning column')
446 Mymodel.materials['Steel leaning column'].Elastic(table=((Q/F)*E, pois), ))
447 Mymodel.materials['Steel leaning column'].Plastic(table=((Q/F)*fy, 0.0), ((Q/F)*fu, strain)))
448
449 #geometry partition adjustment
450 #part 1
451 Mypart1.PartitionEdgeByPoint(edge=Mypart1.edges.findAt((0.0, 0.0, 0.0), ), point=Mypart1.
452 InterestingPoint(Mypart1.edges.findAt((0.0, 0.0, 0.0), ), MIDDLE))
453
454 #part 4
455 Mypart4.PartitionEdgeByPoint(edge=Mypart4.edges.findAt((b1+b2, 0.0, 0.0), ), point=Mypart4.
456 InterestingPoint(Mypart4.edges.findAt((b1+b2, 0.0, 0.0), ), MIDDLE))
457
458 #sets
459 #set 1
460 if (joint ==1):
461 Myassembly.Set(name='top_nodes', referencePoints=(Myassembly.referencePoints[32], Myassembly.
462 referencePoints[33], Myassembly.referencePoints[34]))
463
464 elif (joint==2):
465 Myassembly.Set(name='top_nodes', referencePoints=(Myassembly.referencePoints[36], Myassembly.
466 referencePoints[37], Myassembly.referencePoints[38]))
467
468 #set 2
469 Myassembly.Set(name='bottom_nodes', vertices=Myassembly.instances['Part-1-1'].vertices.findAt(((0.0,
470 0.0, 0.0), ), )+Myassembly.instances['Part-1-2'].vertices.findAt(((b1, 0.0, 0.0), ), )+Myassembly.
471 instances['Part-4-1'].vertices.findAt(((b1+b2, 0.0, 0.0), ), ))
472
473 #interactions and couplings

```

```

437         #coupling column A
438     Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-1-1'].vertices.findAt(((0.0,
0.0, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name=
'Constraint-13', surface=Region(edges=Myassembly.instances['Part-1-1'].edges.findAt((-Plh/2, 0.0, -
Plb/4), ), ((Plh/2, 0.0, -Plb/4), ), ((Plh/2, 0.0, Plb/4), ), ((-Plh/2, 0.0, Plb/4), ), ((-Plh/4, 0.0
, 0.0), ), ((Plh/4, 0.0, 0.0), ), )), u1=ON, u2=ON, u3=ON, ur1=ON, ur2=ON, ur3=ON)
439     #coupling column B
440     Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-1-2'].vertices.findAt(((b1,
0.0, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name=
'Constraint-14', surface=Region(edges=Myassembly.instances['Part-1-2'].edges.findAt(((b1-Plh/2, 0.0,
-Plb/4), ), ((b1+Plh/2, 0.0, -Plb/4), ), ((b1+Plh/2, 0.0, Plb/4), ), ((b1-Plh/2, 0.0, Plb/4), ), ((b1
-Plh/4, 0.0, 0.0), ), ((b1+Plh/4, 0.0, 0.0), ), )), u1=ON, u2=ON, u3=ON, ur1=ON, ur2=ON, ur3=ON)
441     #coupling column C
442     Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-4-1'].vertices.findAt(((b1+
b2, 0.0, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name=
'Constraint-15', surface=Region(edges=Myassembly.instances['Part-4-1'].edges.findAt(((b1+b2-P4h/2,
0.0, -P4b/4), ), ((b1+b2+P4h/2, 0.0, -P4b/4), ), ((b1+b2+P4h/2, 0.0, P4b/4), ), ((b1+b2-P4h/2, 0.0,
P4b/4), ), ((b1+b2-P4h/4, 0.0, 0.0), ), ((b1+b2+P4h/4, 0.0, 0.0), ), )), u1=ON, u2=ON, u3=ON, ur1=ON,
ur2=ON, ur3=ON)
443
444     #boundary condition
445     #stabilizing frame
446     if (condition == 1):
447         Mymodel.PinnedBC(createStepName='Initial', localCsys=None, name='BC-1', region=Region(vertices=
Myassembly.instances['Part-1-1'].vertices.findAt(((0.0, 0.0, 0.0), ), )+Myassembly.instances[
'Part-1-2'].vertices.findAt(((b1, 0.0, 0.0), ), ))
448     elif (condition == 2):
449         Mymodel.EncastreBC(createStepName='Initial', localCsys=None, name='BC-1', region=Region(vertices=
Myassembly.instances['Part-1-1'].vertices.findAt(((0.0, 0.0, 0.0), ), )+Myassembly.instances[
'Part-1-2'].vertices.findAt(((b1, 0.0, 0.0), ), ))
450     #leaning column
451     Mymodel.PinnedBC(createStepName='Initial', localCsys=None, name='BC-2', region=Region(vertices=
Myassembly.instances['Part-4-1'].vertices.findAt(((b1+b2, 0.0, 0.0), )))
452     #z-direction
453     Mymodel.DisplacementBC(amplitude=UNSET, createStepName='Initial', distributionType=UNIFORM, fieldName
='', localCsys=None, name='BC-3', region=Region(faces=Myassembly.instances['Part-1-1'].faces.findAt
(((Plh/4, h, 0.0), (0.0, 0.0, 1.0)), ((-Plh/4, h, 0.0), (0.0, 0.0, 1.0)), ((0.0, h/4, 0.0), (0.0, 0.0
, 1.0)), )+Myassembly.instances['Part-2-1'].faces.findAt(((b1/2, h, 0.0), (0.0, 0.0, 1.0)), )+
Myassembly.instances['Part-1-2'].faces.findAt(((b1+Plh/4, h, 0.0), (0.0, 0.0, 1.0)), ((b1-Plh/4, h,
0.0), (0.0, 0.0, 1.0)), ((b1, h/4, 0.0), (0.0, 0.0, 1.0)), )+Myassembly.instances['Part-5-1'].faces.
findAt(((b1+b2/2, h, 0.0), (0.0, 0.0, 1.0)), )+Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2+
P4h/4, h, 0.0), (0.0, 0.0, 1.0)), ((b1+b2-P4h/4, h, 0.0), (0.0, 0.0, 1.0)), ((b1+b2, h/4, 0.0), (0.0,
0.0, 1.0)), )), u1=UNSET, u2=UNSET, u3=SET, ur1=UNSET, ur2=UNSET, ur3=UNSET)
454
455     #loads
456     if (H == 0):
457         Mymodel.StaticRiksStep(initialArcInc=0.1, maxNumInc=200, maxArcInc=22, minArcInc=1e-20, name=
'Step-1', nlgeom=ON, previous='Initial', totalArcLength=20.0)
458
459         #vertical load
460         if (joint == 1):
461             Mymodel.ConcentratedForce(cf2=-F, createStepName='Step-1', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_F', region=Region(referencePoints=(Myassembly.referencePoints[32
], Myassembly.referencePoints[33], )))
462             Mymodel.ConcentratedForce(cf2=-Q, createStepName='Step-1', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_Q', region=Region(referencePoints=(Myassembly.referencePoints[34
], )))
463         elif (joint == 2):
464             Mymodel.ConcentratedForce(cf2=-F, createStepName='Step-1', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_F', region=Region(referencePoints=(Myassembly.referencePoints[36
], Myassembly.referencePoints[37], )))
465             Mymodel.ConcentratedForce(cf2=-Q, createStepName='Step-1', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_Q', region=Region(referencePoints=(Myassembly.referencePoints[38
], )))
466
467     else:
468         Mymodel.StaticStep(name='Step-1', previous='Initial')
469         Mymodel.steps['Step-1'].setValues(nlgeom=ON)
470         Mymodel.steps['Step-1'].setValues(minInc=1e-12)
471         Mymodel.StaticRiksStep(initialArcInc=0.1, maxNumInc=200, maxArcInc=22, minArcInc=1e-20, name=
'Step-2', nlgeom=ON, previous='Step-1', totalArcLength=20.0)
472         #horizontal load
473         Mymodel.ConcentratedForce(cf1=-H, createStepName='Step-1', distributionType=UNIFORM, field='',
localCsys=None, name='HOR_H', region=Region(referencePoints=(Myassembly.referencePoints[34], )))
474         #vertical load
475         if (joint == 1):
476             Mymodel.ConcentratedForce(cf2=-F, createStepName='Step-2', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_F', region=Region(referencePoints=(Myassembly.referencePoints[32
], Myassembly.referencePoints[33], )))
477             Mymodel.ConcentratedForce(cf2=-Q, createStepName='Step-2', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_Q', region=Region(referencePoints=(Myassembly.referencePoints[34
], )))
478         elif (joint == 2):
479             Mymodel.ConcentratedForce(cf2=-F, createStepName='Step-2', distributionType=UNIFORM, field=''
, localCsys=None, name='VERT_F', region=Region(referencePoints=(Myassembly.referencePoints[36
], Myassembly.referencePoints[37], )))

```

```

480     Mymodel.ConcentratedForce(cf2=-Q, createStepName='Step-2', distributionType=UNIFORM, field='
      , localCsys=None, name='VERT_Q', region=Region(referencePoints=(Myassembly.referencePoints[38
      1, )))
481
482 #mesh
483     #part 1
484     #y-direction
485     #top
486     Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/2, h,
      0.0), ), ((-P1h/2, h, P1b/2), ), ((P1h/2, h, 0.0), ), ((-P1h/2, h, -P1b/2), ), ((P1h/2, h, P1b/2),
      ), ((P1h/2, h, -P1b/2), ), ), size=P2sy)
487     #bottom
488     Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/2, h/
      4, 0.0), ), ((-P1h/2, h/4, -P1b/2), ), ((P1h/2, h/4, 0.0), ), ((P1h/2, h/4, -P1b/2), ), ((P1h/2, h/4,
      P1b/2), ), ((-P1h/2, h/4, P1b/2), ), ), size=P1sy)
489     #x-direction
490     Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/4, h-
      P2h/2, 0.0), ), ((P1h/4, h+P2h/2, 0.0), ), ((-P1h/4, 0.0, 0.0), ), ((P1h/4, 0.0, 0.0), ), ), size=
      P1sx)
491     #z-direction
492     Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/2, h-
      P2h/2, -P1b/4), ), ((-P1h/2, 0.0, -P1b/4), ), ((-P1h/2, h-P2h/2, P1b/4), ), ((-P1h/2, h+P2h/2, P1b/4
      ), ), ((P1h/2, h-P2h/2, -P1b/4), ), ((P1h/2, 0.0, -P1b/4), ), ((P1h/2, h-P2h/2, P1b/4), ), ((P1h/2,
      0.0, P1b/4), ), ((-P1h/2, 0.0, P1b/4), ), ((-P1h/2, h+P2h/2, -P1b/4), ), ((P1h/2, h+P2h/2, P1b/4), ),
      ((P1h/2, h+P2h/2, -P1b/4), ), ), size=P1sz)
493     #mesh settings
494     Mypart1.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart1.faces.getByBoundingBox(-P1h/2, 0.0,
      -P1b/2, P1h/2, h+P2h/2, P1b/2), technique=FREE)
495     Mypart1.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=OFF
      ), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart1.faces.getByBoundingBox(-P1h/2, 0.0,
      -P1b/2, P1h/2, h+P2h/2, P1b/2), ))
496     Mypart1.generateMesh()
497     #part 2
498     #y-direction
499     Mypart2.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart2.edges.findAt(((b1-P1h/2,
      h, 0.0), ), ((P1h/2, h, 0.0), ), ), size=P2sy)
500     #x-direction
501     Mypart2.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart2.edges.findAt(((b1/4, h+
      P2h/2, 0.0), ), ((b1/4, h+P2h/2, P2b/2), ), ((b1/4, h+P2h/2, -P2b/2), ), ((b1/4, h-P2h/2, 0.0), ), ((
      b1/4, h-P2h/2, P2b/2), ), ((b1/4, h-P2h/2, -P2b/2), ), ), size=P2sx)
502     #z-direction
503     Mypart2.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart2.edges.findAt(((b1-P1h/2,
      h+P2h/2, P2b/4), ), ((P1h/2, h+P2h/2, P2b/4), ), ((P1h/2, h+P2h/2, -P2b/4), ), ((b1-P1h/2, h+P2h/2, -
      P2b/4), ), ((b1-P1h/2, h-P2h/2, P2b/4), ), ((P1h/2, h-P2h/2, P2b/4), ), ((P1h/2, h-P2h/2, -P2b/4), ),
      ((b1-P1h/2, h-P2h/2, -P2b/4), ), ((b1-P1h/2, h+P2h/2, P2b/8), ), ((P1h/2, h+P2h/2, P2b/8), ), ((P1h/
      2, h+P2h/2, -P2b/8), ), ((b1-P1h/2, h+P2h/2, -P2b/8), ), ((b1-P1h/2, h-P2h/2, P2b/8), ), ((P1h/2, h-
      P2h/2, P2b/8), ), ((P1h/2, h-P2h/2, -P2b/8), ), ((b1-P1h/2, h-P2h/2, -P2b/8), ), ), size=P2sz)
504     #mesh settings
505     Mypart2.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart2.faces.getByBoundingBox(P1h/2, h-P2h
      /2, -P2b/2, b1-P1h/2, h+P2h/2, P2b/2), technique=FREE)
506     Mypart2.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=OFF
      ), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart2.faces.getByBoundingBox(P1h/2, h-P2h
      /2, -P2b/2, b1-P1h/2, h+P2h/2, P2b/2), ))
507     Mypart2.generateMesh()
508
509     #part 3a
510     #y-direction
511
512     #x-direction
513     Mypart3a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3a.edges.findAt(((P1h/4,
      0.0, P1b), ), ((P1h/4, 0.0, 0.0), ), ((P1h/4, 0.0, P1b/2), ), ), minSizeFactor=0.1, size=P3sx)
514     #z-direction
515     Mypart3a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3a.edges.findAt(((0.0, 0.0
      , P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4*3), ), ),
      minSizeFactor=0.1, size=P3sz)
516     #mesh settings
517     Mypart3a.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart3a.faces.getByBoundingBox(0.0, 0.0,
      0.0, P1h, 0.0, P1b), technique=FREE)
518     Mypart3a.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=
      OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart3a.faces.getByBoundingBox(0.0, 0.0
      , 0.0, P1h, 0.0, P1b), ))
519     Mypart3a.generateMesh()
520
521     #part 3b
522     #y-direction
523
524     #x-direction
525     Mypart3b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3b.edges.findAt(((P1h/4,
      0.0, P1b), ), ((P1h/4, 0.0, 0.0), ), ((P1h/4, 0.0, P1b/2), ), ), minSizeFactor=0.1, size=P3sx)
526     #z-direction
527     Mypart3b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3b.edges.findAt(((0.0, 0.0
      , P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4*3), ), ),
      minSizeFactor=0.1, size=P3sz)
528     #mesh settings
529     Mypart3b.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart3b.faces.getByBoundingBox(0.0, 0.0,
      0.0, P1h, 0.0, P1b), technique=FREE)

```

```

530 Mypart3b.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=
OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart3b.faces.getByBoundingBox(0.0, 0.0
, 0.0, P1h, 0.0, P1b), ))
531 Mypart3b.generateMesh()
532
533 #part 4
534 #y-direction
535 #top
536 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
2, h, 0.0), ), ((b1+b2-P4h/2, h, P4b/2), ), ((b1+b2+P4h/2, h, 0.0), ), ((b1+b2-P4h/2, h, -P4b/2), ),
((b1+b2+P4h/2, h, P4b/2), ), ((b1+b2+P4h/2, h, -P4b/2), ), ), size=P5sy)
537 #bottom
538 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
2, h/4, 0.0), ), ((b1+b2-P4h/2, h/4, -P4b/2), ), ((b1+b2+P4h/2, h/4, 0.0), ), ((b1+b2+P4h/2, h/4, -
P4b/2), ), ((b1+b2+P4h/2, h/4, P4b/2), ), ((b1+b2-P4h/2, h/4, P4b/2), ), ), size=P4sy)
539 #x-direction
540 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
4, h-P5h/2, 0.0), ), ((b1+b2+P4h/4, h+P5h/2, 0.0), ), ((b1+b2-P4h/4, 0.0, 0.0), ), ((b1+b2+P4h/4, 0.0
, 0.0), ), ), size=P4sx)
541 #z-direction
542 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
2, h-P5h/2, -P4b/4), ), ((b1+b2-P4h/2, 0.0, -P4b/4), ), ((b1+b2-P4h/2, h-P5h/2, P4b/4), ), ((b1+b2-
P4h/2, h+P5h/2, P4b/4), ), ((b1+b2+P4h/2, h-P5h/2, -P4b/4), ), ((b1+b2+P4h/2, 0.0, -P4b/4), ), ((b1+
b2+P4h/2, h-P5h/2, P4b/4), ), ((b1+b2+P4h/2, 0.0, P4b/4), ), ((b1+b2-P4h/2, 0.0, P4b/4), ), ((b1+b2-
P4h/2, h+P5h/2, -P4b/4), ), ((b1+b2+P4h/2, h+P5h/2, P4b/4), ), ((b1+b2+P4h/2, h+P5h/2, -P4b/4), ), ),
size=P4sz)
543 #mesh settings
544 Mypart4.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart4.faces.getByBoundingBox(b1+b2-P4h/2,
0.0, -P4b/2, b1+b2+P4h/2, h+P5h/2, P4b/2), technique=FREE)
545 Mypart4.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=OFF)
), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart4.faces.getByBoundingBox(b1+b2-P4h/2,
0.0, -P4b/2, b1+b2+P4h/2, h+P5h/2, P4b/2), ))
546 Mypart4.generateMesh()
547
548 #part 5
549 if (joint ==1):
550 #y-direction
551 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h+P5h/4, 0.0), ), ((b1+b2-P4h/2, h-P5h/4, 0.0), ), ((b1+P1h/2, h+P5h/4, 0.0), ), ((b1+P1h/2
, h-P5h/4, 0.0), ), ), size=P5sy)
552 #x-direction
553 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2/
4, h+P5h/2, 0.0), ), ((b1+b2/4, h+P5h/2, P5b/2), ), ((b1+b2/4, h+P5h/2, -P5b/2), ), ((b1+b2/4, h-
P5h/2, 0.0), ), ((b1+b2/4, h-P5h/2, P5b/2), ), ((b1+b2/4, h-P5h/2, -P5b/2), ), ), size=P5sx)
554 #z-direction
555 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, -P5b/4), ), ((b1+
b2-P4h/2, h+P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, P5b/4), ), ((b1+P1h/2, h-P5h/2, P5b/4), ),
((b1+P1h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h+P5h/2, P5b
/8), ), ((b1+P1h/2, h+P5h/2, P5b/8), ), ((b1+P1h/2, h+P5h/2, -P5b/8), ), ((b1+b2-P4h/2, h+P5h/2,
-P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/
2, -P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/8), ), ), size=P5sz)
556 #mesh settings
557 Mypart5.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart5.faces.getByBoundingBox(b1+P1h/2
, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), technique=FREE)
558 Mypart5.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy
=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart5.faces.getByBoundingBox(b1+
P1h/2, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), ))
559 Mypart5.generateMesh()
560
561 if (joint ==2):
562 #y-direction
563 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h, 0.0), ), ((b1+P1h/2, h+P5h/4, 0.0), ), ((b1+P1h/2, h-P5h/4, 0.0), ), ), size=P5sy)
564 #x-direction
565 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2/
4, h+P5h/2, 0.0), ), ((b1+b2/4, h+P5h/2, P5b/2), ), ((b1+b2/4, h+P5h/2, -P5b/2), ), ((b1+b2/4, h-
P5h/2, 0.0), ), ((b1+b2/4, h-P5h/2, P5b/2), ), ((b1+b2/4, h-P5h/2, -P5b/2), ), ), size=P5sx)
566 #z-direction
567 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, -P5b/4), ), ((b1+
b2-P4h/2, h+P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, P5b/4), ), ((b1+P1h/2, h-P5h/2, P5b/4), ),
((b1+P1h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h+P5h/2, P5b
/8), ), ((b1+P1h/2, h+P5h/2, P5b/8), ), ((b1+P1h/2, h+P5h/2, -P5b/8), ), ((b1+b2-P4h/2, h+P5h/2,
-P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/
2, -P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/8), ), ), size=P5sz)
568 #mesh settings
569 Mypart5.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart5.faces.getByBoundingBox(b1+P1h/2
, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), technique=FREE)
570 Mypart5.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy
=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart5.faces.getByBoundingBox(b1+
P1h/2, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), ))
571 Mypart5.generateMesh()
572
573
574 if (joint ==1):

```

```

575         #part 6a
576             #y-direction
577
578             #x-direction
579 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((P4h/
580 4, 0.0, P4b), ), ((P4h/4, 0.0, 0.0), ), ((P4h/4, 0.0, P4b/2), ), ), size=P6sx)
581             #z-direction
582 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((0.0,
583 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4*3), ), ), size
584 =P6sz)
585             #mesh settings
586 Mypart6a.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart6a.faces.getByBoundingBox(0.0,
587 0.0, 0.0, P4h, 0.0, P4b), technique=FREE)
588 Mypart6a.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD,
589 secondOrderAccuracy=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart6a.faces.
590 getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b), ))
591 Mypart6a.generateMesh()
592 elif (joint ==2):
593     #part 6a
594         #y-direction
595
596         #x-direction
597 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((P4h/
598 4, 0.0, P4b), ), ((P4h/4, 0.0, 0.0), ), ((P4h/4, 0.0, P4b/2), ), ), size=P6sx)
599         #z-direction
600 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((0.0,
601 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4*3), ), ), size
602 =P6sz)
603         #mesh settings
604 Mypart6a.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart6a.faces.getByBoundingBox(0.0,
605 0.0, 0.0, P4h, 0.0, P4b), technique=FREE)
606 Mypart6a.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD,
607 secondOrderAccuracy=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart6a.faces.
608 getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b), ))
609 Mypart6a.generateMesh()
610
611     #part 6b
612         #y-direction
613
614         #x-direction
615 Mypart6b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6b.edges.findAt(((P4h/
616 4, 0.0, P4b), ), ((P4h/4, 0.0, 0.0), ), ((P4h/4, 0.0, P4b/2), ), ), size=P6sx)
617         #z-direction
618 Mypart6b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6b.edges.findAt(((0.0,
619 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4*3), ), ), size
620 =P6sz)
621         #mesh settings
622 Mypart6b.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart6b.faces.getByBoundingBox(0.0,
623 0.0, 0.0, P4h, 0.0, P4b), technique=FREE)
624 Mypart6b.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD,
625 secondOrderAccuracy=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart6b.faces.
626 getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b), ))
627 Mypart6b.generateMesh()
628
629 #element set
630
631 #####error#####
632 Myassembly.Set(elements=Myassembly.instances['Part-1-1'].elements[0:348]+Myassembly.instances[
633 'Part-2-1'].elements[0:312]+Myassembly.instances['Part-1-2'].elements[0:348]+Myassembly.instances[
634 'Part-3a-1'].elements[0:16]+Myassembly.instances['Part-3b-1'].elements[0:16]+Myassembly.instances[
635 'Part-3b-1'].elements[0:16]+Myassembly.instances['Part-3b-2'].elements[0:16], name='Set-11')
636 Myassembly.Set(elements=Myassembly.instances['Part-1-2'].elements[283:284], name='Set-11')
637 #####error#####
638
639 #node set
640 Myassembly.Set(name='LE', nodes=Myassembly.instances['Part-1-1'].nodes[337:338])
641
642 #flanges
643 for lnf in [1, 2, 3, 4, 5, 6]:
644     #lnf = 6
645     Myassembly.Set(elements=
646         Myassembly.instances['Part-1-1'].elements.getByBoundingBox(-Plh/2, 0.0, -(lnf*Plsz)-1, -Plh/2
647 , h+P2h/2, -((lnf-1)*Plsz)+1)+Myassembly.instances['Part-1-1'].elements.getByBoundingBox(Plh/
648 2, 0.0, -(lnf*Plsz)-1, Plh/2, h+P2h/2, -((lnf-1)*Plsz)+1)+Myassembly.instances['Part-1-1'].
649 elements.getByBoundingBox(-Plh/2, 0.0, (lnf-1)*Plsz-1, -Plh/2, h+P2h/2, (lnf)*Plsz+1)+
650 Myassembly.instances['Part-1-1'].elements.getByBoundingBox(Plh/2, 0.0, (lnf-1)*Plsz-1, Plh/2,
651 h+P2h/2, lnf*Plsz+1)
652         +Myassembly.instances['Part-1-2'].elements.getByBoundingBox(b1-Plh/2, 0.0, -(lnf*Plsz)-1, b1-
653 Plh/2, h+P2h/2, -(lnf-1)*Plsz+1)+Myassembly.instances['Part-1-2'].elements.getByBoundingBox(
654 b1+Plh/2, 0.0, -(lnf*Plsz)-1, b1+Plh/2, h+P2h/2, -(lnf-1)*Plsz+1)+Myassembly.instances[
655 'Part-1-2'].elements.getByBoundingBox(b1-Plh/2, 0.0, (lnf-1)*Plsz-1, b1-Plh/2, h+P2h/2, (lnf*
656 Plsz)+1)+Myassembly.instances['Part-1-2'].elements.getByBoundingBox(b1+Plh/2, 0.0, (lnf-1)*
657 Plsz-1, b1+Plh/2, h+P2h/2, (lnf*Plsz)+1)
658         , name='stress_P1F_'+'{0}'.format(lnf))
659     Myassembly.Set(elements=
660         Myassembly.instances['Part-2-1'].elements.getByBoundingBox(Plh/2, h+P2h/2, -lnf*P2sz-1, b1-
661 Plh/2, h+P2h/2, -(lnf-1)*P2sz+1)+Myassembly.instances['Part-2-1'].elements.getByBoundingBox(

```

```

        Plh/2, h-P2h/2, (lnf-1)*P2sz-1, b1-Plh/2, h-P2h/2, lnf*P2sz+1)+Myassembly.instances[
        'Part-2-1'].elements.getByBoundingBox(Plh/2, h-P2h/2, -lnf*P2sz-1, b1-Plh/2, h-P2h/2, -(lnf-1)*
        )*P2sz+1)+Myassembly.instances['Part-2-1'].elements.getByBoundingBox(Plh/2, h+P2h/2, (lnf-1)*
        P2sz-1, b1-Plh/2, h+P2h/2, lnf*P2sz+1)
629     , name='stress_P2F_'+'{0}'.format(lnf))
630
631 Myassembly.Set(elements=
        Myassembly.instances['Part-4-1'].elements.getByBoundingBox(b1+b2-P4h/2, 0.0, -lnf*P4sz-1, b1+
        b2-P4h/2, h+P5h/2, -(lnf-1)*P4sz+1)+Myassembly.instances['Part-4-1'].elements.
        getByBoundingBox(b1+b2+P4h/2, 0.0, -lnf*P4sz-1, b1+b2+P4h/2, h+P5h/2, -(lnf-1)*P4sz+1)+
        Myassembly.instances['Part-4-1'].elements.getByBoundingBox(b1+b2-P4h/2, 0.0, (lnf-1)*P4sz-1,
        b1+b2-P4h/2, h+P5h/2, lnf*P4sz+1)+Myassembly.instances['Part-4-1'].elements.getByBoundingBox(
        b1+b2+P4h/2, 0.0, (lnf-1)*P4sz-1, b1+b2+P4h/2, h+P5h/2, lnf*P4sz+1)
632     , name='stress_P4F_'+'{0}'.format(lnf))
633
634 Myassembly.Set(elements=
        Myassembly.instances['Part-5-1'].elements.getByBoundingBox(b1+Plh/2, h+P5h/2, -lnf*P5sz-1, b1
        +b2-P4h/2, h+P5h/2, -(lnf-1)*P5sz+1)+Myassembly.instances['Part-5-1'].elements.
        getByBoundingBox(b1+Plh/2, h-P5h/2, (lnf-1)*P5sz-1, b1+b2-P4h/2, h-P5h/2, lnf*P5sz+1)+
        Myassembly.instances['Part-5-1'].elements.getByBoundingBox(b1+Plh/2, h-P5h/2, -lnf*P5sz-1, b1
        +b2-P4h/2, h-P5h/2, -(lnf-1)*P5sz+1)+Myassembly.instances['Part-5-1'].elements.
        getByBoundingBox(b1+Plh/2, h+P5h/2, (lnf-1)*P5sz-1, b1+b2-P4h/2, h+P5h/2, lnf*P5sz+1)
635     , name='stress_P5F_'+'{0}'.format(lnf))
636
637     #web
638 for lnw in [1, 2, 3, 4]:
639     Myassembly.Set(elements=
640         Myassembly.instances['Part-1-1'].elements.getByBoundingBox(-lnw*Plsx-1, 0.0, 0.0, -(lnw-1)*
        Plsx+1, h+P2h/2, 0.0)+Myassembly.instances['Part-1-1'].elements.getByBoundingBox((lnw-1)*Plsx
        -1, 0.0, 0.0, lnw*Plsx+1, h+P2h/2, 0.0)
641         +Myassembly.instances['Part-1-2'].elements.getByBoundingBox(b1-lnw*Plsx, 0.0, 0.0, b1-(lnw-1)
        )*Plsx, h+P2h/2, 0.0)+Myassembly.instances['Part-1-2'].elements.getByBoundingBox(b1+(lnw-1)*
        Plsx, 0.0, 0.0, b1+lnw*Plsx, h+P2h/2, 0.0)
642         , name='stress_P1W_'+'{0}'.format(lnw))
643
644     Myassembly.Set(elements=
        Myassembly.instances['Part-2-1'].elements.getByBoundingBox(Plh/2, h+(lnw-1)*P2sy, 0.0, b1-Plh
        /2, h+lnw*P2sy, 0.0)+Myassembly.instances['Part-2-1'].elements.getByBoundingBox(Plh/2, h-lnw*
        P2sy, 0.0, b1-Plh/2, h-(lnw-1)*P2sy, 0.0)
645         , name='stress_P2W_'+'{0}'.format(lnw))
646
647     Myassembly.Set(elements=
        Myassembly.instances['Part-4-1'].elements.getByBoundingBox(b1+b2-lnw*P4sx, 0.0, 0.0, b1+b2-(
        lnw-1)*P4sx, h+P5h/2, 0.0)+Myassembly.instances['Part-4-1'].elements.getByBoundingBox(b1+b2+(
        lnw-1)*P4sx, 0.0, 0.0, b1+b2+lnw*P4sx, h+P5h/2, 0.0)
648         , name='stress_P4W_'+'{0}'.format(lnw))
649
650     Myassembly.Set(elements=
        Myassembly.instances['Part-5-1'].elements.getByBoundingBox(b1+Plh/2, h+(lnw-1)*P5sy, 0.0, b1+
        b2-P4h/2, h+lnw*P5sy, 0.0)+Myassembly.instances['Part-5-1'].elements.getByBoundingBox(b1+Plh/
        2, h-lnw*P5sy, 0.0, b1+b2-P4h/2, h-(lnw-1)*P5sy, 0.0)
651         , name='stress_P5W_'+'{0}'.format(lnw))
652
653     #total
654 if (joint ==1):
655     Myassembly.Set(name='frame_node', nodes=
656         Myassembly.instances['Part-1-1'].nodes.getByBoundingBox(-Plh/2, 0.0, -Plb/2, Plh/2, h+P2h/2,
        Plb/2)+
657         Myassembly.instances['Part-2-1'].nodes.getByBoundingBox(Plh/2, h-P2h/2, -P2b/2, b1-Plh/2, h+
        P2h/2, P2b/2)+
658         Myassembly.instances['Part-1-2'].nodes.getByBoundingBox(b1-Plh/2, 0.0, -Plb/2, b1+Plh/2, h+
        P2h/2, Plb/2)+
659         Myassembly.instances['Part-3a-1'].nodes.getByBoundingBox(-Plh/2, h+P2h/2, -Plb/2, Plh/2, h+
        P2h/2, Plb/2)+
660         Myassembly.instances['Part-3b-1'].nodes.getByBoundingBox(-Plh/2, h-P2h/2, -Plb/2, Plh/2, h-
        P2h/2, Plb/2)+
661         Myassembly.instances['Part-3a-2'].nodes.getByBoundingBox(b1-Plh/2, h+P2h/2, -Plb/2, b1+Plh/2,
        h+P2h/2, Plb/2)+
662         Myassembly.instances['Part-3b-2'].nodes.getByBoundingBox(b1-Plh/2, h-P2h/2, -Plb/2, b1+Plh/2,
        h-P2h/2, Plb/2)+
663         Myassembly.instances['Part-4-1'].nodes.getByBoundingBox(b1+b2-P4h/2, 0.0, -P4b/2, b1+b2+P4h/2
        , h+P5h/2, P4b/2)+
664         Myassembly.instances['Part-5-1'].nodes.getByBoundingBox(b1+Plh/2, h-P5h/2, -P5b/2, b1+b2-P4h/
        2, h+P5h/2, P5b/2)+
665         Myassembly.instances['Part-6a-1'].nodes.getByBoundingBox(b1+b2-P4h/2, h+P5h/2, -P4b/2, b1+b2+
        P4h/2, h+P5h/2, P4b/2))
666
667 elif (joint==2):
668     Myassembly.Set(name='frame_node', nodes=
669         Myassembly.instances['Part-1-1'].nodes.getByBoundingBox(-Plh/2, 0.0, -Plb/2, Plh/2, h+P2h/2,
        Plb/2)+
670         Myassembly.instances['Part-2-1'].nodes.getByBoundingBox(Plh/2, h-P2h/2, -P2b/2, b1-Plh/2, h+
        P2h/2, P2b/2)+
671         Myassembly.instances['Part-1-2'].nodes.getByBoundingBox(b1-Plh/2, 0.0, -Plb/2, b1+Plh/2, h+
        P2h/2, Plb/2)+
672         Myassembly.instances['Part-3a-1'].nodes.getByBoundingBox(-Plh/2, h+P2h/2, -Plb/2, Plh/2, h+
        P2h/2, Plb/2)+
673         Myassembly.instances['Part-3b-1'].nodes.getByBoundingBox(-Plh/2, h-P2h/2, -Plb/2, Plh/2, h-
        P2h/2, Plb/2)+
674         Myassembly.instances['Part-3a-2'].nodes.getByBoundingBox(b1-Plh/2, h+P2h/2, -Plb/2, b1+Plh/2,
        h+P2h/2, Plb/2)+

```



```

675 Myassembly.instances['Part-3b-2'].nodes.getByBoundingBox(b1-P1h/2, h-P2h/2, -P1b/2, b1+P1h/2,
676 h-P2h/2, P1b/2)+
677 Myassembly.instances['Part-4-1'].nodes.getByBoundingBox(b1+b2-P4h/2, 0.0, -P4b/2, b1+b2+P4h/2
678 , h+P5h/2, P4b/2)+
679 Myassembly.instances['Part-5-1'].nodes.getByBoundingBox(b1+P1h/2, h-P5h/2, -P5b/2, b1+b2-P4h/
680 2, h+P5h/2, P5b/2)+
681 Myassembly.instances['Part-6a-1'].nodes.getByBoundingBox(b1+b2-P4h/2, h+P5h/2, -P4b/2, b1+b2+
682 P4h/2, h+P5h/2, P4b/2)+
683 Myassembly.instances['Part-6b-1'].nodes.getByBoundingBox(b1+b2-P4h/2, h-P5h/2, -P4b/2, b1+b2+
684 P4h/2, h-P5h/2, P4b/2))
685
686 #field output request
687 Mymodel.fieldOutputRequests['F-Output-1'].setValues(variables=('S', 'E', 'PE', 'PEEQ', 'PEMAG', 'LE',
688 'U', 'RF', 'CF', 'CSTRESS', 'CDISP'))
689
690 #history Output Requests and applying the imperfection
691 if (H ==0):
692     del Mymodel.historyOutputRequests['H-Output-1']
693     Mymodel.HistoryOutputRequest(createStepName='Step-1', name='U1', rebar=EXCLUDE, region=Myassembly
694     .sets['top_nodes'], sectionPoints=DEFAULT, variables=('U1', ))
695     Mymodel.HistoryOutputRequest(createStepName='Step-1', name='RF2', rebar=EXCLUDE, region=
696     Myassembly.sets['bottom_nodes'], sectionPoints=DEFAULT, variables=('RF2', ))
697     #Load applied
698     Mymodel.HistoryOutputRequest(createStepName='Step-1', name='CF2', rebar=EXCLUDE, region=
699     Myassembly.sets['top_nodes'], sectionPoints=DEFAULT, variables=('CF2', ))
700     #Strain
701     Mymodel.HistoryOutputRequest(createStepName='Step-1', name='LE', rebar=EXCLUDE, region=Myassembly
702     .sets['LE'], sectionPoints=DEFAULT, variables=('LE11', 'LE22', 'LE33', 'LE12', 'LE13', 'LE23',
703     'LEP'))
704
705     Mymodel.keywordBlock.synchVersions(storeNodesAndElements=False)
706     sentence = 348
707     if (A2 == 0):
708         Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-"+'{0}'.format(id)+"",
709         STEP=1, NSET=frame_node\n"+'{0}'.format(model)+"", "+'{0}'.format(A1))
710     elif (A3 == 0):
711         Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-"+'{0}'.format(id)+"",
712         STEP=1, NSET=frame_node\n"+'{0}'.format(model)+"", "+'{0}'.format(A1)+"\n"+'{0}'.format(mode2
713         )+", "+'{0}'.format(A2))
714     else:
715         Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-"+'{0}'.format(id)+"",
716         STEP=1, NSET=frame_node\n"+'{0}'.format(model)+"", "+'{0}'.format(A1)+"\n"+'{0}'.format(mode2
717         )+", "+'{0}'.format(A2)+"\n"+'{0}'.format(mode3)+"", "+'{0}'.format(A3))
718
719     else:
720         Mymodel.HistoryOutputRequest(createStepName='Step-2', name='U1', rebar=EXCLUDE, region=Myassembly
721         .sets['top_nodes'], sectionPoints=DEFAULT, variables=('U1', ))
722         Mymodel.HistoryOutputRequest(createStepName='Step-2', name='RF2', rebar=EXCLUDE, region=
723         Myassembly.sets['bottom_nodes'], sectionPoints=DEFAULT, variables=('RF2', ))
724         #Load applied
725         Mymodel.HistoryOutputRequest(createStepName='Step-2', name='CF2', rebar=EXCLUDE, region=
726         Myassembly.sets['top_nodes'], sectionPoints=DEFAULT, variables=('CF2', ))
727         #Strain
728         Mymodel.HistoryOutputRequest(createStepName='Step-2', name='LE', rebar=EXCLUDE, region=Myassembly
729         .sets['LE'], sectionPoints=DEFAULT, variables=('LE11', 'LE22', 'LE33', 'LE12', 'LE13', 'LE23',
730         'LEP'))
731
732         Mymodel.keywordBlock.synchVersions(storeNodesAndElements=False)
733         sentence = 349
734         if (A2 == 0):
735             Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-"+'{0}'.format(id)+"",
736             STEP=1, NSET=frame_node\n"+'{0}'.format(model)+"", "+'{0}'.format(A1))
737         elif (A3 == 0):
738             Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-"+'{0}'.format(id)+"",
739             STEP=1, NSET=frame_node\n"+'{0}'.format(model)+"", "+'{0}'.format(A1)+"\n"+'{0}'.format(mode2
740             )+", "+'{0}'.format(A2))
741         else:
742             Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-"+'{0}'.format(id)+"",
743             STEP=1, NSET=frame_node\n"+'{0}'.format(model)+"", "+'{0}'.format(A1)+"\n"+'{0}'.format(mode2
744             )+", "+'{0}'.format(A2)+"\n"+'{0}'.format(mode3)+"", "+'{0}'.format(A3))
745
746 #applying residual stresses
747 for lnf in [1, 2, 3]:
748     residual_stress = ((fy/6*5)-(fy/3*(lnf-1)))*rs1
749     Mymodel.keywordBlock.insert(sentence+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P1F_'+'{0}'
750     .format(lnf)+'', '+'{0}'.format(residual_stress))
751 for lnf in [4, 5, 6]:
752     residual_stress = -(fy/6)-(fy/3*(lnf-4))*rs1
753     Mymodel.keywordBlock.insert(sentence+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P1F_'+'{0}'
754     .format(lnf)+'', '+'{0}'.format(residual_stress))
755 for lnf in [1, 2, 3]:
756     residual_stress = ((fy/6*5)-(fy/3*(lnf-1)))*rs2
757     Mymodel.keywordBlock.insert(sentence+6+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P2F_'+
758     '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
759 for lnf in [4, 5, 6]:
760     residual_stress = -(fy/6)-(fy/3*(lnf-4))*rs2
761     Mymodel.keywordBlock.insert(sentence+6+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P2F_'+

```

```

    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
732 for lnf in [1, 2, 3]:
733     residual_stress = ((fy/6*5)-(fy/3*(lnf-1)))*rs4
734     Mymodel.keywordBlock.insert(sentence+12+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P4F_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
735 for lnf in [4, 5, 6]:
736     residual_stress = -(fy/6)-(fy/3*(lnf-4))*rs4
737     Mymodel.keywordBlock.insert(sentence+12+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P4F_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
738 for lnf in [1, 2, 3]:
739     residual_stress = ((fy/6*5)-(fy/3*(lnf-1)))*rs5
740     Mymodel.keywordBlock.insert(sentence+18+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P5F_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
741 for lnf in [4, 5, 6]:
742     residual_stress = -(fy/6)-(fy/3*(lnf-4))*rs5
743     Mymodel.keywordBlock.insert(sentence+18+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P5F_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
744
745 for lnf in [1, 2]:
746     residual_stress = -(fy/4*3)+(fy/2*(lnf-1))*rs1
747     Mymodel.keywordBlock.insert(sentence+24+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P1W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
748 for lnf in [3, 4]:
749     residual_stress = ((fy/4)+(fy/2*(lnf-3)))*rs1
750     Mymodel.keywordBlock.insert(sentence+24+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P1W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
751 for lnf in [1, 2]:
752     residual_stress = -(fy/4*3)+(fy/2*(lnf-1))*rs2
753     Mymodel.keywordBlock.insert(sentence+28+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P2W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
754 for lnf in [3, 4]:
755     residual_stress = ((fy/4)+(fy/2*(lnf-3)))*rs2
756     Mymodel.keywordBlock.insert(sentence+28+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P2W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
757 for lnf in [1, 2]:
758     residual_stress = -(fy/4*3)+(fy/2*(lnf-1))*rs4
759     Mymodel.keywordBlock.insert(sentence+32+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P4W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
760 for lnf in [3, 4]:
761     residual_stress = ((fy/4)+(fy/2*(lnf-3)))*rs4
762     Mymodel.keywordBlock.insert(sentence+32+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P4W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
763 for lnf in [1, 2]:
764     residual_stress = -(fy/4*3)+(fy/2*(lnf-1))*rs5
765     Mymodel.keywordBlock.insert(sentence+36+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P5W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
766 for lnf in [3, 4]:
767     residual_stress = ((fy/4)+(fy/2*(lnf-3)))*rs5
768     Mymodel.keywordBlock.insert(sentence+36+lnf, '\n*INITIAL CONDITIONS, TYPE=STRESS\nstress_P5W_'+
    '{0}'.format(lnf)+'', '+'{0}'.format(residual_stress))
769
770 #job
771 mdb.Job(atTime=None, contactPrint=OFF, description='', echoPrint=OFF, explicitPrecision=SINGLE,
getMemoryFromAnalysis=True, historyPrint=OFF, memory=90, memoryUnits=PERCENTAGE, model=
'GMNIA_IV-frame-'+id, modelPrint=OFF, multiprocessingMode=DEFAULT, name='GMNIA_IV-job-'+id,
nodalOutputPrecision=SINGLE, numCpus=1, numGPUs=0, queue=None, resultsFormat=ODB, scratch='', type=
ANALYSIS, userSubroutine='', waitHours=0, waitMinutes=0)
772
773 mdb.jobs['GMNIA_IV-job-'+id].submit(consistencyChecking=OFF)
774 mdb.jobs['GMNIA_IV-job-'+id].waitForCompletion()
775
776 #Create ODB output
777 if (H== 0):
778     RF_A, RF_B, RF_C, U1_A, U1_B, U1_C, CF_A, CF_B, CF_C, = ([ for i in range (9)
779
780     Directory_odb = '{0}'.format(Loadpath)+'/'+'GMNIA_IV-job-'+id+'.odb'
781     open_odb = session.openOdb(name=Directory_odb)
782     odb = session.odbs[Directory_odb]
783     session.viewports['Viewport: 1'].setValues(displayedObject=open_odb)
784     frames = open_odb.steps['Step-1'].frames
785     numFrames = len(frames)
786
787     #Reaction Force
788     xy_RF_A = session.XYDataFromHistory(name='XY-RF_A', odb=odb, outputVariableName='Reaction force:
RF2 PI: PART-1-1 Node 19', steps=('Step-1', ), )
789     RF_A = [abs(x[1]*10**(-3)) for x in xy_RF_A]
790     Max_RF_A = max(RF_A)
791
792     xy_RF_B = session.XYDataFromHistory(name='XY-RF_B', odb=odb, outputVariableName='Reaction force:
RF2 PI: PART-1-2 Node 19', steps=('Step-1', ), )
793     RF_B = [abs(x[1]*10**(-3)) for x in xy_RF_B]
794     Max_RF_B = max(RF_B)
795
796     xy_RF_C = session.XYDataFromHistory(name='XY-RF_C', odb=odb, outputVariableName='Reaction force:
RF2 PI: PART-4-1 Node 19', steps=('Step-1', ), )
797     RF_C = [abs(x[1]*10**(-3)) for x in xy_RF_C]

```

```

798     Max_RF_C = max(RF_C)
799
800     #Displacement
801     xy_U1_A = session.XYDataFromHistory(name='XY_U', odb=odb, outputVariableName='Spatial
displacement: U1 PI: rootAssembly Node 1 in NSET TOP_NODES', steps=('Step-1', ), )
802     U1_A = [abs(x[1]*-1) for x in xy_U1_A]
803
804     xy_U1_B = session.XYDataFromHistory(name='XY_U', odb=odb, outputVariableName='Spatial
displacement: U1 PI: rootAssembly Node 2 in NSET TOP_NODES', steps=('Step-1', ), )
805     U1_B = [abs(x[1]*-1) for x in xy_U1_A]
806
807     xy_U1_C = session.XYDataFromHistory(name='XY_U', odb=odb, outputVariableName='Spatial
displacement: U1 PI: rootAssembly Node 3 in NSET TOP_NODES', steps=('Step-1', ), )
808     U1_C = [abs(x[1]*-1) for x in xy_U1_A]
809
810     #Applied Force
811     xy_CF_A = session.XYDataFromHistory(name='XY-CF_A', odb=odb, outputVariableName='Point loads:
CF2 PI: rootAssembly Node 1 in NSET TOP_NODES', steps=('Step-1', ), )
812     CF_A = [abs(x[1]*10**(-3)) for x in xy_CF_A]
813     Max_CF_A = max(CF_A)
814
815     xy_CF_B = session.XYDataFromHistory(name='XY-CF_B', odb=odb, outputVariableName='Point loads:
CF2 PI: rootAssembly Node 2 in NSET TOP_NODES', steps=('Step-1', ), )
816     CF_B = [abs(x[1]*10**(-3)) for x in xy_CF_B]
817     Max_CF_B = max(CF_B)
818
819     xy_CF_C = session.XYDataFromHistory(name='XY-CF_C', odb=odb, outputVariableName='Point loads:
CF2 PI: rootAssembly Node 3 in NSET TOP_NODES', steps=('Step-1', ), )
820     CF_C = [abs(x[1]*10**(-3)) for x in xy_CF_C]
821     Max_CF_C = max(CF_C)
822
823     #CREATE EXCEL VALUES
824     INC=range(0, numFrames)
825
826     workbook = xlswriter.Workbook('GMNIA_IV-'+'{0}'.format(Filename))
827     worksheet1 = workbook.add_worksheet('ABAQUSDATA')
828
829     #Write general data
830     bold = workbook.add_format({'bold': 1})
831     headings1 = ['','{0}'.format(Filename)]
832     headings2 = ['INC', 'U1_A', 'RF_A', 'CF_A', 'U1_B', 'RF_B', 'CF_B', 'U1_C', 'RF_C', 'CF_C']
833     headings3 = ['(-)', '(mm)', '(kN)', '(kN)', '(mm)', '(kN)', '(kN)', '(mm)', '(kN)', '(kN)']
834     headings4 = ['', 'Max_RF_A', 'Max_CF_A', 'Max_RF_B', 'Max_CF_B', 'Max_RF_C', 'Max_CF_C']
835     worksheet1.write_row('A1', headings1, bold)
836     worksheet1.write_row('A2', headings2, bold)
837     worksheet1.write_row('A3', headings3)
838     worksheet1.write_column(3, 0, INC)
839     worksheet1.write_column(3, 1, U1_A)
840     worksheet1.write_column(3, 2, RF_A)
841     worksheet1.write_column(3, 3, CF_A)
842     worksheet1.write_column(3, 4, U1_B)
843     worksheet1.write_column(3, 5, RF_B)
844     worksheet1.write_column(3, 6, CF_B)
845     worksheet1.write_column(3, 7, U1_C)
846     worksheet1.write_column(3, 8, RF_C)
847     worksheet1.write_column(3, 9, CF_C)
848     worksheet1.write_row('A205', headings4, bold)
849
850     workbook.close()
851
852     else:
853         RF_A, RF_B, RF_C, U1_A, U1_B, U1_C, CF_A, CF_B, CF_C, = ([ for i in range (9))
854
855         Directory_odb = '{0}'.format(Loadpath)+'/'+'GMNIA_IV-job-'+id+'.odb'
856         open_odb = session.openOdb(name=Directory_odb)
857         odb = session.odbs[Directory_odb]
858         session.viewports['Viewport: 1'].setValues(displayedObject=open_odb)
859         frames = open_odb.steps['Step-2'].frames
860         numFrames = len(frames)
861
862         #Reaction Force
863         xy_RF_A = session.XYDataFromHistory(name='XY-RF_A', odb=odb, outputVariableName='Reaction force:
RF2 PI: PART-1-1 Node 19', steps=('Step-2', ), )
864         RF_A = [abs(x[1]*10**(-3)) for x in xy_RF_A]
865         Max_RF_A = max(RF_A)
866
867         xy_RF_B = session.XYDataFromHistory(name='XY-RF_B', odb=odb, outputVariableName='Reaction force:
RF2 PI: PART-1-2 Node 19', steps=('Step-2', ), )
868         RF_B = [abs(x[1]*10**(-3)) for x in xy_RF_B]
869         Max_RF_B = max(RF_B)
870
871         xy_RF_C = session.XYDataFromHistory(name='XY-RF_C', odb=odb, outputVariableName='Reaction force:
RF2 PI: PART-4-1 Node 19', steps=('Step-2', ), )
872         RF_C = [abs(x[1]*10**(-3)) for x in xy_RF_C]
873         Max_RF_C = max(RF_C)
874

```

```

875     #Displacement
876     xy_U1_A = session.XYDataFromHistory(name='XY_U', odb=odb, outputVariableName='Spatial
displacement: U1 PI: rootAssembly Node 1 in NSET TOP_NODES', steps=('Step-2', ), )
877     U1_A = [abs(x[1]*-1) for x in xy_U1_A]
878
879     xy_U1_B = session.XYDataFromHistory(name='XY_U', odb=odb, outputVariableName='Spatial
displacement: U1 PI: rootAssembly Node 2 in NSET TOP_NODES', steps=('Step-2', ), )
880     U1_B = [abs(x[1]*-1) for x in xy_U1_A]
881
882     xy_U1_C = session.XYDataFromHistory(name='XY_U', odb=odb, outputVariableName='Spatial
displacement: U1 PI: rootAssembly Node 3 in NSET TOP_NODES', steps=('Step-2', ), )
883     U1_C = [abs(x[1]*-1) for x in xy_U1_A]
884
885     #Applied Force
886     xy_CF_A = session.XYDataFromHistory(name='XY-CF_A', odb=odb, outputVariableName='Point loads:
CF2 PI: rootAssembly Node 1 in NSET TOP_NODES', steps=('Step-2', ), )
887     CF_A = [abs(x[1]*10**(-3)) for x in xy_CF_A]
888     Max_CF_A = max(CF_A)
889
890     xy_CF_B = session.XYDataFromHistory(name='XY-CF_B', odb=odb, outputVariableName='Point loads:
CF2 PI: rootAssembly Node 2 in NSET TOP_NODES', steps=('Step-2', ), )
891     CF_B = [abs(x[1]*10**(-3)) for x in xy_CF_B]
892     Max_CF_B = max(CF_B)
893
894     xy_CF_C = session.XYDataFromHistory(name='XY-CF_C', odb=odb, outputVariableName='Point loads:
CF2 PI: rootAssembly Node 3 in NSET TOP_NODES', steps=('Step-2', ), )
895     CF_C = [abs(x[1]*10**(-3)) for x in xy_CF_C]
896     Max_CF_C = max(CF_C)
897
898     #CREATE EXCEL VALUES
899     INC=range(0, numFrames)
900
901     workbook = xlswriter.Workbook('GMNIA_IV-'+'{0}.xlsx'.format(Filename))
902     worksheet1 = workbook.add_worksheet('ABAQUSDATA')
903
904     #Write general data
905     bold = workbook.add_format({'bold': 1})
906     headings1 = ['','{0}'.format(Filename)]
907     headings2 = ['INC','U1_A','RF_A','CF_A','U1_B','RF_B','CF_B','U1_C','RF_C','CF_C']
908     headings3 = ['(-)', '(mm)', '(kN)', '(kN)', '(mm)', '(kN)', '(kN)', '(mm)', '(kN)', '(kN)']
909     headings4 = ['','Max_RF_A,Max_CF_A','Max_RF_B,Max_CF_B','Max_RF_C,Max_CF_C']
910     worksheet1.write_row('A1', headings1, bold)
911     worksheet1.write_row('A2', headings2, bold)
912     worksheet1.write_row('A3', headings3)
913     worksheet1.write_column(3, 0, INC)
914     worksheet1.write_column(3, 1, U1_A)
915     worksheet1.write_column(3, 2, RF_A)
916     worksheet1.write_column(3, 3, CF_A)
917     worksheet1.write_column(3, 4, U1_B)
918     worksheet1.write_column(3, 5, RF_B)
919     worksheet1.write_column(3, 6, CF_B)
920     worksheet1.write_column(3, 7, U1_C)
921     worksheet1.write_column(3, 8, RF_C)
922     worksheet1.write_column(3, 9, CF_C)
923     worksheet1.write_row('A205', headings4, bold)
924
925     workbook.close()
926

```

```
1 #GMNIA III is the exact script as GMNIA IV, only the scale factor for the eigenmode and the amount of
2 eigenmodes is different. The corresponding lines that differ are given below:
3
4     e = float(extracted_list[118])
5
6     Mymodel.keywordBlock.synchVersions(storeNodesAndElements=False)
7     sentence = 348
8     Mymodel.keywordBlock.insert(sentence, "\n*IMPERFECTION, FILE=LBA-job-'{0}'.format(id)+",
9     STEP=1, NSET=frame_node\n1, "+'{0}'.format(e))
```

```

1  # -*- coding: mbc8 -*-
2  from part import *
3  from material import *
4  from section import *
5  from assembly import *
6  from step import *
7  from interaction import *
8  from load import *
9  from mesh import *
10 from optimization import *
11 from job import *
12 from sketch import *
13 from visualization import *
14 from connectorBehavior import *
15 from caeModules import *
16 import os
17 import sys
18 import csv
19 import odbAccess
20 import math
21 import numpy as np
22 import xlswriter
23
24 session.journalOptions.setValues(replayGeometry=COORDINATE, recoverGeometry=COORDINATE)
25 # =====setting up=====
26
27 input_file=open('C:/Users/20174572/Google Drive/Wim Broeks/TUe/2. Master/Afstuderen/4. FE Modelling/4.
Parametric/INPUT.txt')
28
29 for line in input_file:
30     extracted_line = line
31     extracted_list = extracted_line.split()
32     if "id" not in extracted_line:
33         id = extracted_list[0]
34         E = float(extracted_list[1])
35         pois = float(extracted_list[2])
36         F = float(extracted_list[3])
37         Q = float(extracted_list[4])
38         H = float(extracted_list[5])
39         fy = float(extracted_list[6])
40         fu = float(extracted_list[7])
41         strain = float(extracted_list[8])
42         h = float(extracted_list[9])
43         b1 = float(extracted_list[10])
44         b2 = float(extracted_list[11])
45         P1h = float(extracted_list[16])
46         P1b = float(extracted_list[17])
47         P1tw = float(extracted_list[18])
48         P1tf = float(extracted_list[19])
49         P2h = float(extracted_list[27])
50         P2b = float(extracted_list[28])
51         P2tw = float(extracted_list[29])
52         P2tf = float(extracted_list[30])
53         P4h = float(extracted_list[38])
54         P4b = float(extracted_list[39])
55         P4tw = float(extracted_list[40])
56         P4tf = float(extracted_list[41])
57         P5h = float(extracted_list[49])
58         P5b = float(extracted_list[50])
59         P5tw = float(extracted_list[51])
60         P5tf = float(extracted_list[52])
61         joint = float(extracted_list[59])
62         condition = float(extracted_list[60])
63         ma1 = float(extracted_list[62])
64         mb1 = float(extracted_list[63])
65         mc1 = float(extracted_list[64])
66         rs1 = float(extracted_list[65])
67         ma2 = float(extracted_list[66])
68         mb2 = float(extracted_list[67])
69         mc2 = float(extracted_list[68])
70         rs2 = float(extracted_list[69])
71         ma4 = float(extracted_list[70])
72         mb4 = float(extracted_list[71])
73         mc4 = float(extracted_list[72])
74         rs4 = float(extracted_list[73])
75         ma5 = float(extracted_list[74])
76         mb5 = float(extracted_list[75])
77         mc5 = float(extracted_list[76])
78         rs5 = float(extracted_list[77])
79
80         #mesh
81         P1sy = ma1
82         P1sx = mb1
83         P1sz = mc1
84
85         P2sy = mb2

```

```

86         P2sx = ma2
87         P2sz = mc2
88
89         P3sx = mb1
90         P3sz = mc1
91
92         P4sy = ma4
93         P4sx = mb4
94         P4sz = mc4
95
96         P5sy = mb5
97         P5sx = ma5
98         P5sz = mc5
99
100        P6sx = mb4
101        P6sz = mc4
102        # =====setting up=====
103        Filename='Frame-'+id
104        Loadpath='F:\\Graduation project Wim\\parametric study\\plot\\{0}'.format(Filename)
105        if not os.path.exists(Loadpath):
106            os.makedirs(Loadpath)
107        os.chdir(Loadpath)
108
109        Mymodel=mdb.Model(name='LBA-frame-'+id)
110        Myassembly=Mymodel.rootAssembly
111
112        #part 1 - stabilizing columns
113        Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
114        Mysketchs=Mymodel.sketches['__sweep__']
115        Mysketchs.Line(point1=(0.0, 0.0), point2=(0.0, h+P2h/2))
116        Mysketchs.geometry.findAt((0.0, h/2))
117        Mysketchs.VerticalConstraint(addUndoState=False, entity=Mysketchs.geometry.findAt((0.0, h/2), ))
118        Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(1.0, 0.0, 0.0, 0.0, 0.0,
119        1.0, -0.0, -1.0, -0.0, 0.0, 0.0, 0.0))
120        Mysketchp=Mymodel.sketches['__profile__']
121        Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
122        Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
123        Mysketchp.Line(point1=(-(P1h/2), (P1b/2)), point2=(-(P1h/2), -(P1b/2)))
124        Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[4])
125        Mysketchp.Line(point1=((P1h/2), (P1b/2)), point2=((P1h/2), -(P1b/2)))
126        Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[5])
127        Mysketchp.Line(point1=(-(P1h/2), 0.0), point2=((P1h/2), 0.0))
128        Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry[6])
129        Mymodel.Part(dimensionality=THREE_D, name='Part-1', type=DEFORMABLE_BODY)
130        Mypart1=Mymodel.parts['Part-1']
131        Mypart1.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
132
133        Mypart1.DatumPlaneByPrincipalPlane(offset=(h-(P2h/2)), principalPlane=XZPLANE)
134        Mypart1.DatumPlaneByPrincipalPlane(offset=(h+(P2h/2)), principalPlane=XZPLANE)
135        Mypart1.DatumPlaneByPrincipalPlane(offset=(h), principalPlane=XZPLANE)
136
137        #part 2 - stabilizing beam
138        Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
139        Mysketchs=Mymodel.sketches['__sweep__']
140        Mysketchs.Line(point1=(P1h/2, h), point2=((b1-P1h/2), h))
141        Mysketchs.HorizontalConstraint(addUndoState=False, entity=Mysketchs.geometry[2])
142        Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(0.0, -1.0, 0.0, -0.0, 0.0,
143        1.0, -1.0, -0.0, -0.0, 0, h, 0.0))
144        Mysketchp=Mymodel.sketches['__profile__']
145        Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
146        Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
147        Mysketchp.Line(point1=(-(P2h/2), (P2b/2)), point2=(-(P2h/2), -(P2b/2)))
148        Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[4])
149        Mysketchp.Line(point1=((P2h/2), (P2b/2)), point2=((P2h/2), -(P2b/2)))
150        Mysketchp.VerticalConstraint(addUndoState=False, entity=Mysketchp.geometry[5])
151        Mysketchp.Line(point1=(-(P2h/2), 0.0), point2=((P2h/2), 0.0))
152        Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry[6])
153        Mymodel.Part(dimensionality=THREE_D, name='Part-2', type=DEFORMABLE_BODY)
154        Mypart2=Mymodel.parts['Part-2']
155        Mypart2.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
156
157        #part 3a - stiffeners stabilizing column
158        Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
159        Mysketchp=Mymodel.sketches['__profile__']
160        Mysketchp.Line(point1=(0.0, 0.0), point2=(P1h, 0.0))
161        Mysketchp.geometry.findAt((P1h/2, 0.0))
162        Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P1h/2, 0.0), ))
163        Mymodel.Part(dimensionality=THREE_D, name='Part-3a', type=DEFORMABLE_BODY)
164        Mypart3a=Mymodel.parts['Part-3a']
165        Mypart3a.BaseShellExtrude(depth=P1b, sketch=Mysketchp)
166        Mypart3a.DatumPlaneByPrincipalPlane(offset=(P1b/2), principalPlane=XYPLANE)
167
168        #part 3b - stiffeners stabilizing column
169        Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
170        Mysketchp=Mymodel.sketches['__profile__']
171        Mysketchp.Line(point1=(0.0, 0.0), point2=(P1h, 0.0))

```

```

170 Mysketchp.geometry.findAt((P1h/2, 0.0))
171 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P1h/2, 0.0), ))
172 Mymodel.Part(dimensionality=THREE_D, name='Part-3b', type=DEFORMABLE_BODY)
173 Mypart3b=Mymodel.parts['Part-3b']
174 Mypart3b.BaseShellExtrude(depth=P1b, sketch=Mysketchp)
175 Mypart3b.DatumPlaneByPrincipalPlane(offset=(P1b/2), principalPlane=XYPLANE)
176
177 #part 4 - leaning column
178 Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
179 Mysketchs=Mymodel.sketches['__sweep__']
180 Mysketchs.Line(point1=(b1+b2, 0.0), point2=(b1+b2, h+P5h/2))
181 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(1.0, 0.0, 0.0, 0.0, 0.0,
182 1.0, -0.0, -1.0, -0.0, b1+b2, 0.0, 0.0))
183 Mysketchp=Mymodel.sketches['__profile__']
184 Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
185 Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
186 Mysketchp.Line(point1=(-P4h/2, P4b/2), point2=(-P4h/2, -P4b/2))
187 Mysketchp.Line(point1=(P4h/2, P4b/2), point2=(P4h/2, -P4b/2))
188 Mysketchp.Line(point1=(-P4h/2, 0.0), point2=(P4h/2, 0.0))
189 Mymodel.Part(dimensionality=THREE_D, name='Part-4', type=DEFORMABLE_BODY)
190 Mypart4=Mymodel.parts['Part-4']
191 Mypart4.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
192
193 Mypart4.DatumPlaneByPrincipalPlane(offset=h-P5h/2, principalPlane=XZPLANE)
194 Mypart4.DatumPlaneByPrincipalPlane(offset=h+P5h/2, principalPlane=XZPLANE)
195 Mypart4.DatumPlaneByPrincipalPlane(offset=(h), principalPlane=XZPLANE)
196
197 #part 5 - leaning beam
198 Mymodel.ConstrainedSketch(name='__sweep__', sheetSize=200.0)
199 Mysketchs=Mymodel.sketches['__sweep__']
200 Mysketchs.Line(point1=(b1+P1h/2, h), point2=(b1+b2-P4h/2, h))
201 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0, transform=(0.0, -1.0, 0.0, -0.0, 0.0,
202 1.0, -1.0, -0.0, -0.0, b1, h, 0.0))
203 Mysketchp=Mymodel.sketches['__profile__']
204 Mysketchp.ConstructionLine(point1=(-100.0, 0.0), point2=(100.0, 0.0))
205 Mysketchp.ConstructionLine(point1=(0.0, -100.0), point2=(0.0, 100.0))
206 Mysketchp.Line(point1=(-P5h/2, P5b/2), point2=(-P5h/2, -P5b/2))
207 Mysketchp.Line(point1=(P5h/2, P5b/2), point2=(P5h/2, -P5b/2))
208 Mysketchp.Line(point1=(-P5h/2, 0.0), point2=(P5h/2, 0.0))
209 Mymodel.Part(dimensionality=THREE_D, name='Part-5', type=DEFORMABLE_BODY)
210 Mypart5=Mymodel.parts['Part-5']
211 Mypart5.BaseShellSweep(path=Mysketchs, sketch=Mysketchp)
212
213 if (joint ==1):
214 #part 6a - stiffeners leaning column
215 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
216 Mysketchp=Mymodel.sketches['__profile__']
217 Mysketchp.Line(point1=(0.0, 0.0), point2=(P4h, 0.0))
218 Mysketchp.geometry.findAt((P4h/2, 0.0))
219 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P4h/2, 0.0),
220 ))
221 Mymodel.Part(dimensionality=THREE_D, name='Part-6a', type=DEFORMABLE_BODY)
222 Mypart6a=Mymodel.parts['Part-6a']
223 Mypart6a.BaseShellExtrude(depth=P4b, sketch=Mysketchp)
224 Mypart6a.DatumPlaneByPrincipalPlane(offset=(P4b/2), principalPlane=XYPLANE)
225
226 elif (joint ==2):
227 #part 6a - stiffeners leaning column
228 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
229 Mysketchp=Mymodel.sketches['__profile__']
230 Mysketchp.Line(point1=(0.0, 0.0), point2=(P4h, 0.0))
231 Mysketchp.geometry.findAt((P4h/2, 0.0))
232 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P4h/2, 0.0),
233 ))
234 Mymodel.Part(dimensionality=THREE_D, name='Part-6a', type=DEFORMABLE_BODY)
235 Mypart6a=Mymodel.parts['Part-6a']
236 Mypart6a.BaseShellExtrude(depth=P4b, sketch=Mysketchp)
237 Mypart6a.DatumPlaneByPrincipalPlane(offset=(P4b/2), principalPlane=XYPLANE)
238
239 #part 6b - stiffeners leaning column
240 Mymodel.ConstrainedSketch(name='__profile__', sheetSize=200.0)
241 Mysketchp=Mymodel.sketches['__profile__']
242 Mysketchp.Line(point1=(0.0, 0.0), point2=(P4h, 0.0))
243 Mysketchp.geometry.findAt((P4h/2, 0.0))
244 Mysketchp.HorizontalConstraint(addUndoState=False, entity=Mysketchp.geometry.findAt((P4h/2, 0.0),
245 ))
246 Mymodel.Part(dimensionality=THREE_D, name='Part-6b', type=DEFORMABLE_BODY)
247 Mypart6b=Mymodel.parts['Part-6b']
248 Mypart6b.BaseShellExtrude(depth=P4b, sketch=Mysketchp)
249 Mypart6b.DatumPlaneByPrincipalPlane(offset=(P4b/2), principalPlane=XYPLANE)
250
251 #assembly
252 Myassembly.DatumCsysByDefault(CARTESIAN)
253 Myassembly.Instance(dependent=ON, name='Part-1-1', part=Mypart1)
254 Myassembly.Instance(dependent=ON, name='Part-2-1', part=Mypart2)
255 Myassembly.Instance(dependent=ON, name='Part-1-2', part=Mypart1)
256 Myassembly.translate(instanceList=('Part-1-2', ), vector=(b1, 0.0, 0.0))
257 Myassembly.Instance(dependent=ON, name='Part-4-1', part=Mypart4)

```



```

251 Myassembly.Instance(dependent=ON, name='Part-5-1', part=Mypart5)
252
253 #partition
254 #part 1
255 Mypart1.PartitionFaceByDatumPlane(datumPlane=Mypart1.datums[2], faces=Mypart1.faces.findAt(((P1h/2,
h/4, P1b/4), ), ((-P1h/2, h/4, -P1b/4), ), ((P1h/2, h/4, P1b/4), ), ((P1h/2, h/4, -P1b/4), ), ((P1h/4
, h/4, 0.0), ), ))
256 #part 4
257 Mypart4.PartitionFaceByDatumPlane(datumPlane=Mypart4.datums[2], faces=Mypart4.faces.findAt(((b1+b2-
P4h/2, h/4, P4b/4), ), ((b1+b2-P4h/2, h/4, -P4b/4), ), ((b1+b2+P4h/2, h/4, P4b/4), ), ((b1+b2+P4h/2,
h/4, -P4b/4), ), ((b1+b2, h/4, 0.0), ), ))
258
259 #part 3a-1
260 Mypart3a.PartitionFaceByDatumPlane(datumPlane=Mypart3a.datums[2], faces=Mypart3a.faces.findAt(((P1h/4
, 0.0, P1b/4), ))
261 Myassembly.Instance(dependent=ON, name='Part-3a-1', part=Mypart3a)
262 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-1-1'].edges.findAt(((P1h/2, h
+(P2h/2), P1b/4), ), ((P1h/2, h+(P2h/2), P1b/4), ), ((-P1h/2, h+(P2h/2), -P1b/4), ), ((P1h/2, h
+(P2h/2), -P1b/4), ), ((0.0, h+(P2h/2), 0.0), ), ), name='Tie_A_a', positionToleranceMethod=COMPUTED,
slave=Region(edges=Myassembly.instances['Part-3a-1'].edges.findAt(((0.0, 0.0, P1b/4), ), ((P1h, 0.0,
P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h/2, 0.0, P1b/2), ), ), ), thickness=
ON, tieRotations=ON)
263 Myassembly.translate(instanceList=('Part-3a-1', ), vector=(-P1h/2, h+(P2h/2), -(P1b/2)))
264
265 #part 3b-1
266 Mypart3b.PartitionFaceByDatumPlane(datumPlane=Mypart3b.datums[2], faces=Mypart3b.faces.findAt(((P1h/4
, 0.0, P1b/4), ))
267 Myassembly.Instance(dependent=ON, name='Part-3b-1', part=Mypart3b)
268 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-1-1'].edges.findAt(((P1h/2, h
-(P2h/2), P1b/4), ), ((P1h/2, h-(P2h/2), P1b/4), ), ((-P1h/2, h-(P2h/2), -P1b/4), ), ((P1h/2, h
-(P2h/2), -P1b/4), ), ((0.0, h-(P2h/2), 0.0), ), ), name='Tie_A_b', positionToleranceMethod=
COMPUTED, slave=Region(edges=Myassembly.instances['Part-3b-1'].edges.findAt(((0.0, 0.0, P1b/4), ), ((
P1h, 0.0, P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h/2, 0.0, P1b/2), ), ), ),
thickness=ON, tieRotations=ON)
269 Myassembly.translate(instanceList=('Part-3b-1', ), vector=(-P1h/2, (h-(P2h/2)), -(P1b/2)))
270
271 #part 3a-2
272 Myassembly.Instance(dependent=ON, name='Part-3a-2', part=Mypart3a)
273 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-1-2'].edges.findAt(((P1h/2)+
b1, h+(P2h/2), P1b/4), ), ((P1h/2)+b1, h+(P2h/2), P1b/4), ), ((-P1h/2)+b1, h+(P2h/2), -P1b/4), ),
(((P1h/2)+b1, h+(P2h/2), -P1b/4), ), ((b1, h+(P2h/2), 0.0), ), ), name='Tie_B_a',
positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-3a-2'].edges.findAt
(((0.0, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h
/2, 0.0, P1b/2), ), ), ), thickness=ON, tieRotations=ON)
274 Myassembly.translate(instanceList=('Part-3a-2', ), vector=(-P1h/2+b1, (h+(P2h/2)), -(P1b/2)))
275
276 #part 3b-2
277 Myassembly.Instance(dependent=ON, name='Part-3b-2', part=Mypart3b)
278 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-1-2'].edges.findAt(((P1h/2)+
b1, h-(P2h/2), P1b/4), ), ((P1h/2)+b1, h-(P2h/2), P1b/4), ), ((-P1h/2)+b1, h-(P2h/2), -P1b/4), ),
(((P1h/2)+b1, h-(P2h/2), -P1b/4), ), ((b1, h-(P2h/2), 0.0), ), ), name='Tie_B_b',
positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-3b-2'].edges.findAt
(((0.0, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4*3), ), ((P1h
/2, 0.0, P1b/2), ), ), ), thickness=ON, tieRotations=ON)
279 Myassembly.translate(instanceList=('Part-3b-2', ), vector=(-P1h/2+b1, (h-(P2h/2)), -(P1b/2)))
280
281 if (joint ==1):
282 #part 6-1
283 Mypart6a.PartitionFaceByDatumPlane(datumPlane=Mypart6a.datums[2], faces=Mypart6a.faces.findAt(((
P4h/4, 0.0, P4b/4), ))
284 Myassembly.Instance(dependent=ON, name='Part-6a-1', part=Mypart6a)
285 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-4-1'].edges.findAt(((P4h/
2)+b1+b2, h+(P5h/2), P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), P4b/4), ), ((-P4h/2)+b1+b2, h+(P5h/2
), -P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), -P4b/4), ), ((b1+b2, h+(P5h/2), 0.0), ), ), name=
'Tie_C_a', positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-6a-1'
].edges.findAt(((0.0, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0,
P4b/4*3), ), ((P4h/2, 0.0, P4b/2), ), ), ), thickness=ON, tieRotations=ON)
286 Myassembly.translate(instanceList=('Part-6a-1', ), vector=(-P4h/2+b1+b2, (h+(P5h/2)), -(P4b/2)))
287
288 elif (joint ==2):
289 #part 6-1
290 Mypart6a.PartitionFaceByDatumPlane(datumPlane=Mypart6a.datums[2], faces=Mypart6a.faces.findAt(((
P4h/4, 0.0, P4b/4), ))
291 Myassembly.Instance(dependent=ON, name='Part-6a-1', part=Mypart6a)
292 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-4-1'].edges.findAt(((P4h/
2)+b1+b2, h+(P5h/2), P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), P4b/4), ), ((-P4h/2)+b1+b2, h+(P5h/2
), -P4b/4), ), ((P4h/2)+b1+b2, h+(P5h/2), -P4b/4), ), ((b1+b2, h+(P5h/2), 0.0), ), ), name=
'Tie_C_a', positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-6a-1'
].edges.findAt(((0.0, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0,
P4b/4*3), ), ((P4h/2, 0.0, P4b/2), ), ), ), thickness=ON, tieRotations=ON)
292 Myassembly.translate(instanceList=('Part-6a-1', ), vector=(-P4h/2+b1+b2, (h+(P5h/2)), -(P4b/2)))
293 #part 6-2
294 Mypart6b.PartitionFaceByDatumPlane(datumPlane=Mypart6b.datums[2], faces=Mypart6b.faces.findAt(((
P4h/4, 0.0, P4b/4), ))
295 Myassembly.Instance(dependent=ON, name='Part-6b-1', part=Mypart6b)
296 Mymodel.Tie(adjust=ON, master=Region(edges=Myassembly.instances['Part-4-1'].edges.findAt(((P4h/

```

```

280 2)+b1+b2, h-(P5h/2), P4b/4), ), ((P4h/2)+b1+b2, h-(P5h/2), P4b/4), ), ((-P4h/2)+b1+b2, h-(P5h/2
281 ), -P4b/4), ), ((P4h/2)+b1+b2, h-(P5h/2), -P4b/4), ), ((b1+b2, h-(P5h/2), 0.0), ), ), name=
282 'Tie_C_b', positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-6b-1'
283 ].edges.findAt(((0.0, 0.0, P4b/4), ), (P4h, 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), (P4h, 0.0,
284 P4b/4*3), ), (P4h/2, 0.0, P4b/2), ), ), thickness=ON, tieRotations=ON)
297 Myassembly.translate(instanceList=('Part-6b-1', ), vector=(-P4h/2+b1+b2, (h-(P5h/2)), -(P4b/2)))
298
299 #interaction
300 #reference points
301 Myassembly.ReferencePoint(point=(0.0, h, 0.0))
302 Myassembly.ReferencePoint(point=(b1, h, 0.0))
303 Myassembly.ReferencePoint(point=(b1+b2, h, 0.0))
304 #part 1-1 with 2-1
305 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, 1000.0, 0.0))
306 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-1-1'].faces.findAt(((P1h/2, h,
307 P1b/4), ), ((P1h/2, h, -P1b/4), ), ), name='Constraint-7', positionToleranceMethod=COMPUTED, slave=
308 Region(edges=Myassembly.instances['Part-2-1'].edges.findAt(((P1h/2, h+P2h/2+1000, P2b/4), ), ((P1h/2,
309 h+P2h/2+1000, -P2b/4), ), ((P1h/2, h-P2h/2+1000, P2b/4), ), ((P1h/2, h-P2h/2+1000, -P2b/4), ), ((P1h
310 /2, h+1000, 0.0), ), ), ), thickness=ON, tieRotations=ON)
311 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, -1000.0, 0.0))
312
313 #part 1-2 with 2-1
314 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, 1000.0, 0.0))
315 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-1-2'].faces.findAt(((b1-P1h/2,
316 h, P1b/4), ), ((b1-P1h/2, h, -P1b/4), ), ), name='Constraint-8', positionToleranceMethod=COMPUTED,
317 slave=Region(edges=Myassembly.instances['Part-2-1'].edges.findAt(((b1-P1h/2, h-P2h/2+1000, -P1b/4),
318 ), ((b1-P1h/2, h-P2h/2+1000, P1b/4), ), ((b1-P1h/2, h+1000, 0.0), ), ((b1-P1h/2, h+P2h/2+1000, P1b/4
319 ), ), ((b1-P1h/2, h+P2h/2+1000, -P1b/4), ), ), ), thickness=ON, tieRotations=ON)
320 Myassembly.translate(instanceList=('Part-2-1', ), vector=(0.0, -1000.0, 0.0))
321
322 #part 1-2 with 5-1
323 Mypart5.PartitionEdgeByPoint(edge=Mypart5.edges.findAt((b1+P1h/2, h, 0.0), ), point=Mypart5.
324 InterestingPoint(Mypart5.edges.findAt((b1+P1h/2, h, 0.0), MIDDLE))
325 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, 1000.0, 0.0))
326 Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-5-1'].vertices.findAt(((b1+
327 P1h/2, h+1000, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name
328 ='Constraint-9', surface=Region(edges=Myassembly.instances['Part-5-1'].edges.findAt(((b1+P1h/2, h+P5h
329 /2+1000, P5b/4), ), (b1+P1h/2, h+P5h/2+1000, -P5b/4), ), (b1+P1h/2, h-P5h/2+1000, P5b/4), ), (b1+
330 P1h/2, h-P5h/2+1000, -P5b/4), ), (b1+P1h/2, h+P5h/4+1000, 0.0), ), (b1+P1h/2, h-P5h/4+1000, 0.0),
331 ), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3=ON)
332 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-1-2'].faces.findAt(((b1+P1h/2,
333 h, P1b/4), ), ((b1+P1h/2, h, -P1b/4), ), ), name='Constraint-10', positionToleranceMethod=COMPUTED,
334 slave=Region(vertices=Myassembly.instances['Part-5-1'].vertices.findAt(((b1+P1h/2, h+1000, 0.0), )),
335 thickness=ON, constraintRatio=10000.0, constraintRatioMethod=SPECIFIED, tieRotations=OFF)
336 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, -1000.0, 0.0))
337
338 #part 4-1 with 5-1
339 if (joint ==1):
340 Mypart5.PartitionEdgeByPoint(edge=Mypart5.edges.findAt((b1+b2-P4h/2, h, 0.0), ), point=Mypart5.
341 InterestingPoint(Mypart5.edges.findAt((b1+b2-P4h/2, h, 0.0), MIDDLE))
342 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, 1000.0, 0.0))
343 Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-5-1'].vertices.findAt(((
344 b1+b2-P4h/2, h+1000, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=
345 None, name='Constraint-11', surface=Region(edges=Myassembly.instances['Part-5-1'].edges.findAt(((
346 b1+b2-P4h/2, h+P5h/2+1000, P5b/4), ), ((b1+b2-P4h/2, h+P5h/2+1000, -P5b/4), ), ((b1+b2-P4h/2, h-
347 P5h/2+1000, P5b/4), ), ((b1+b2-P4h/2, h-P5h/2+1000, -P5b/4), ), ((b1+b2-P4h/2, h+P5h/4+1000, 0.0
348 ), ), ((b1+b2-P4h/2, h-P5h/4+1000, 0.0), ), ), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3=ON)
349 Mymodel.Tie(adjust=ON, constraintRatio=10000.0, constraintRatioMethod=SPECIFIED, master=Region(
350 faces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2-P4h/2, h, P4b/4), ), ((b1+b2-P4h/2, h
351 , -P4b/4), ), ), name='Constraint-12', positionToleranceMethod=COMPUTED, slave=Region(vertices=
352 Myassembly.instances['Part-5-1'].vertices.findAt(((b1+b2-P4h/2, h+1000, 0.0), )), thickness=ON,
353 tieRotations=OFF)
354 Myassembly.translate(instanceList=('Part-5-1', ), vector=(0.0, -1000.0, 0.0))
355
356 elif (joint ==2):
357 Mymodel.Tie(adjust=ON, master=Region(faces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2-
358 P4h/2, h, P4b/4), ), ((b1+b2-P4h/2, h, -P4b/4), ), ), name='Constraint-11',
359 positionToleranceMethod=COMPUTED, slave=Region(edges=Myassembly.instances['Part-5-1'].edges.
360 findAt(((b1+b2-P4h/2, h+P5h/2, P5b/4), ), (b1+b2-P4h/2, h+P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-
361 P5h/2, P5b/4), ), ((b1+b2-P4h/2, h, 0.0), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/4), ), ), ), thickness=ON
362 , tieRotations=ON)
363
364 #coupling reference points
365 if (joint ==1):
366 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[32], )),
367 couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-16',
368 surface=Region(sidelFaces=Myassembly.instances['Part-1-1'].faces.findAt(((P1h/4, h, 0.0), (0.0,
369 0.0, 1.0)), ((-P1h/4, h, 0.0), (0.0, 0.0, 1.0)), ), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3=ON)
370 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[33], )),
371 couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-17',
372 surface=Region(sidelFaces=Myassembly.instances['Part-1-2'].faces.findAt(((b1+P1h/4, h, 0.0), (0.0
373 , 0.0, 1.0)), ((b1-P1h/4, h, 0.0), (0.0, 0.0, 1.0)), ), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3
374 =ON)
375 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[34], )),
376 couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-18',
377 surface=Region(sidelFaces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2+P4h/4, h, 0.0), (

```

```

0.0, 0.0, 1.0)), ((b1+b2-P4h/4, h, 0.0), (0.0, 0.0, 1.0)), ), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=
ON, ur3=ON)
336 elif (joint ==2):
337 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[36], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-16',
surface=Region(sidelFaces=Myassembly.instances['Part-1-1'].faces.findAt(((Plh/4, h, 0.0), (0.0,
0.0, 1.0)), ((-Plh/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3=ON)
338 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[37], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-17',
surface=Region(sidelFaces=Myassembly.instances['Part-1-2'].faces.findAt(((b1+Plh/4, h, 0.0), (0.0,
0.0, 1.0)), ((b1-Plh/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=ON, ur3
=ON)
339 Mymodel.Coupling(controlPoint=Region(referencePoints=(Myassembly.referencePoints[38], )),
couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name='Constraint-18',
surface=Region(sidelFaces=Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2+P4h/4, h, 0.0), (
0.0, 0.0, 1.0)), ((b1+b2-P4h/4, h, 0.0), (0.0, 0.0, 1.0))), ), u1=ON, u2=ON, u3=ON, url=ON, ur2=
ON, ur3=ON)
340
341 #cross-sections
342 #part 1
343 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P1tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P1tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
344 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P1tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P1tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
345 #part 2
346 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P2tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P2tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
347 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P2tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P2tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
348 #part 4
349 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel leaning column', name='P4tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF,
temperature=GRADIENT, thickness=P4tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM
, useDensity=OFF)
350 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel leaning column', name='P4tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF,
temperature=GRADIENT, thickness=P4tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM
, useDensity=OFF)
351 #part 5
352 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P5tf', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P5tf, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
353 Mymodel.HomogeneousShellSection(idealization=NO_IDEALIZATION, integrationRule=SIMPSON, material=
'Steel', name='P5tw', numIntPts=5, poissonDefinition=DEFAULT, preIntegrate=OFF, temperature=GRADIENT,
thickness=P5tw, thicknessField='', thicknessModulus=None, thicknessType=UNIFORM, useDensity=OFF)
354
355 #section assignment
356 #part 1
357 #bottom flange
358 Mypart1.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
Mypart1.faces.findAt(((Plh/2, h/4, -Plb/4), ), ((Plh/2, h, Plb/4), ), ((Plh/2, h, -Plb/4), ), ((Plh/2
, h/4, Plb/4), ), ), ), sectionName='P1tf', thicknessAssignment=FROM_SECTION)
359 #top flange
360 Mypart1.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart1.faces.findAt(((Plh/2, h/4, -Plb/4), ), ((-Plh/2, h, Plb/4), ), ((-Plh/2, h, -Plb/4), ), ((-
Plh/2, h/4, Plb/4), ), ), ), sectionName='P1tf', thicknessAssignment=FROM_SECTION)
361 #web
362 Mypart1.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
Mypart1.faces.findAt(((0.0, h/4, 0.0), ), ((0, h, 0.0), ), ), ), sectionName='P1tw',
thicknessAssignment=FROM_SECTION)
363
364 #part 2
365 #bottom flange
366 Mypart2.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
Mypart2.faces.findAt(((b1/2, h-(P2h/2), -P2b/4), ), ((b1/2, h-(P2h/2), P2b/4), ), ), ), sectionName=
'P2tf', thicknessAssignment=FROM_SECTION)
367 #top flange
368 Mypart2.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart2.faces.findAt(((b1/2, h+(P2h/2), -P2b/4), ), ((b1/2, h+(P2h/2), P2b/4), ), ), ), sectionName=
'P2tf', thicknessAssignment=FROM_SECTION)
369 #web
370 Mypart2.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
Mypart2.faces.findAt(((b1/2, h, 0.0), )), ), sectionName='P2tw', thicknessAssignment=FROM_SECTION)
371
372 #part 3a
373 Mypart3a.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart3a.faces.getByBoundingBox(0.0, 0.0, 0.0, Plh, 0.0, Plb)), sectionName='P2tf',
thicknessAssignment=FROM_SECTION)
374 #part 3b
375 Mypart3b.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces
=Mypart3b.faces.getByBoundingBox(0.0, 0.0, 0.0, Plh, 0.0, Plb)), sectionName='P2tf',
thicknessAssignment=FROM_SECTION)

```

```

376
377     #part 4
378     #bottom flange
379     Mypart4.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
Mypart4.faces.findAt(((b1+b2-P4h/2, h/4, -P4b/4), ), ((b1+b2-P4h/2, h, P4b/4), ), ((b1+b2-P4h/2, h/4,
P4b/4), ), ((b1+b2-P4h/2, h, -P4b/4), ), ), ), sectionName='P4tf', thicknessAssignment=FROM_SECTION)

380
381     #top flange
Mypart4.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart4.faces.findAt(((b1+b2+P4h/2, h/4, -P4b/4), ), ((b1+b2+P4h/2, h/4, P4b/4), ), ((b1+b2+P4h/2, h,
P4b/4), ), ((b1+b2+P4h/2, h, -P4b/4), ), ), ), sectionName='P4tf', thicknessAssignment=FROM_SECTION)

382
383     #web
Mypart4.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
Mypart4.faces.findAt(((b1+b2, h, 0.0), ), ((b1+b2, h/4, 0.0), ), ), ), sectionName='P4tw',
thicknessAssignment=FROM_SECTION)

384
385     #part 5
386     #bottom flange
387     Mypart5.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(faces=
Mypart5.faces.findAt(((b1+b2/2, h-P5h/2, P5b/4), ), ((b1+b2/2, h-P5h/2, -P5b/4), ), ), ), sectionName=
'P5tf', thicknessAssignment=FROM_SECTION)
388     #top flange
389     Mypart5.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(faces=
Mypart5.faces.findAt(((b1+b2/2, h+P5h/2, P5b/4), ), ((b1+b2/2, h+P5h/2, -P5b/4), ), ), ), sectionName=
'P5tf', thicknessAssignment=FROM_SECTION)
390     #web
391     Mypart5.SectionAssignment(offset=0.0, offsetField='', offsetType=MIDDLE_SURFACE, region=Region(faces=
Mypart5.faces.findAt(((b1+b2/2, h, 0.0), ))), sectionName='P5tw', thicknessAssignment=FROM_SECTION)

392
393     #part 6a
394     if (joint==1):
395     #part 6a
396     Mypart6a.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(
faces=Mypart6a.faces.getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b)), sectionName='P5tf',
thicknessAssignment=FROM_SECTION)
397     elif (joint ==2):
398     #part 6a
399     Mypart6a.SectionAssignment(offset=0.0, offsetField='', offsetType=TOP_SURFACE, region=Region(
faces=Mypart6a.faces.getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b)), sectionName='P5tf',
thicknessAssignment=FROM_SECTION)
400     #part 6b
401     Mypart6b.SectionAssignment(offset=0.0, offsetField='', offsetType=BOTTOM_SURFACE, region=Region(
faces=Mypart6b.faces.getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b)), sectionName='P5tf',
thicknessAssignment=FROM_SECTION)

402
403     #material
404     Mymodel.Material(name='Steel')
405     Mymodel.materials['Steel'].Elastic(table=((E, pois), ))
406     Mymodel.Material(name='Steel leaning column')
407     Mymodel.materials['Steel leaning column'].Elastic(table=((Q/F)*E, pois), ))
408
409     #geometry partition adjustment
410     #part 1
411     Mypart1.PartitionEdgeByPoint(edge=Mypart1.edges.findAt((0.0, 0.0, 0.0), ), point=Mypart1.
InterestingPoint(Mypart1.edges.findAt((0.0, 0.0, 0.0), ), MIDDLE))

412
413     #part 4
414     Mypart4.PartitionEdgeByPoint(edge=Mypart4.edges.findAt((b1+b2, 0.0, 0.0), ), point=Mypart4.
InterestingPoint(Mypart4.edges.findAt((b1+b2, 0.0, 0.0), ), MIDDLE))

415
416     #sets
417     #set 1
418     if (joint ==1):
419     Myassembly.Set(name='top_nodes', referencePoints=(Myassembly.referencePoints[32], Myassembly.
referencePoints[33], Myassembly.referencePoints[34]))
420     elif (joint==2):
421     Myassembly.Set(name='top_nodes', referencePoints=(Myassembly.referencePoints[36], Myassembly.
referencePoints[37], Myassembly.referencePoints[38]))
422     #set 2
423     Myassembly.Set(name='bottom_nodes', vertices=Myassembly.instances['Part-1-1'].vertices.findAt(((0.0,
0.0, 0.0), ), )+Myassembly.instances['Part-1-2'].vertices.findAt(((b1, 0.0, 0.0), ), )+Myassembly.
instances['Part-4-1'].vertices.findAt(((b1+b2, 0.0, 0.0), ), ))

424
425     #interactions and couplings
426     #coupling column A
427     Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-1-1'].vertices.findAt(((0.0,
0.0, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name=
'Constraint-13', surface=Region(edges=Myassembly.instances['Part-1-1'].edges.findAt(((P1h/2, 0.0, -
P1b/4), ), ((P1h/2, 0.0, -P1b/4), ), ((P1h/2, 0.0, P1b/4), ), ((-P1h/2, 0.0, P1b/4), ), ((-P1h/4, 0.0
, 0.0), ), ((P1h/4, 0.0, 0.0), ), ), ), u1=ON, u2=ON, u3=ON, ur1=ON, ur2=ON, ur3=ON)
428     #coupling column B
429     Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-1-2'].vertices.findAt(((b1,
0.0, 0.0), )), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name=
'Constraint-14', surface=Region(edges=Myassembly.instances['Part-1-2'].edges.findAt(((b1-P1h/2, 0.0,
-P1b/4), ), ((b1+P1h/2, 0.0, -P1b/4), ), ((b1+P1h/2, 0.0, P1b/4), ), ((b1-P1h/2, 0.0, P1b/4), ), ((b1

```

```

-Plh/4, 0.0, 0.0), ), ((b1+Plh/4, 0.0, 0.0), ), ), ), u1=ON, u2=ON, u3=ON, ur1=ON, ur2=ON, ur3=ON)
430 #coupling column C
431 Mymodel.Coupling(controlPoint=Region(vertices=Myassembly.instances['Part-4-1'].vertices.findAt(((b1+
b2, 0.0, 0.0), ))), couplingType=KINEMATIC, influenceRadius=WHOLE_SURFACE, localCsys=None, name=
'Constraint-15', surface=Region(edges=Myassembly.instances['Part-4-1'].edges.findAt(((b1+b2-P4h/2,
0.0, -P4b/4), ), ((b1+b2+P4h/2, 0.0, -P4b/4), ), ((b1+b2+P4h/2, 0.0, P4b/4), ), ((b1+b2-P4h/2, 0.0,
P4b/4), ), ((b1+b2-P4h/4, 0.0, 0.0), ), ((b1+b2+P4h/4, 0.0, 0.0), ), ), ), u1=ON, u2=ON, u3=ON, ur1=ON,
ur2=ON, ur3=ON)
432
433 #boundary condition
434 #stabilizing frame
435 if (condition == 1):
436     Mymodel.PinnedBC(createStepName='Initial', localCsys=None, name='BC-1', region=Region(vertices=
Myassembly.instances['Part-1-1'].vertices.findAt(((0.0, 0.0, 0.0), ), )+Myassembly.instances[
'Part-1-2'].vertices.findAt(((b1, 0.0, 0.0), ), ))
437 elif (condition == 2):
438     Mymodel.EncastreBC(createStepName='Initial', localCsys=None, name='BC-1', region=Region(vertices=
Myassembly.instances['Part-1-1'].vertices.findAt(((0.0, 0.0, 0.0), ), )+Myassembly.instances[
'Part-1-2'].vertices.findAt(((b1, 0.0, 0.0), ), ))
439 #leaning column
440 Mymodel.PinnedBC(createStepName='Initial', localCsys=None, name='BC-2', region=Region(vertices=
Myassembly.instances['Part-4-1'].vertices.findAt(((b1+b2, 0.0, 0.0), )))
441 #Z-direction
442 Mymodel.DisplacementBC(amplitude=UNSET, createStepName='Initial', distributionType=UNIFORM, fieldName=
='', localCsys=None, name='BC-3', region=Region(faces=Myassembly.instances['Part-1-1'].faces.findAt
(((P1h/4, h, 0.0), (0.0, 0.0, 1.0)), ((-P1h/4, h, 0.0), (0.0, 0.0, 1.0)), ((0.0, h/4, 0.0), (0.0, 0.0,
1.0)), )+Myassembly.instances['Part-2-1'].faces.findAt(((b1/2, h, 0.0), (0.0, 0.0, 1.0)), )+
Myassembly.instances['Part-1-2'].faces.findAt(((b1+Plh/4, h, 0.0), (0.0, 0.0, 1.0)), ((b1-P1h/4, h,
0.0), (0.0, 0.0, 1.0)), ((b1, h/4, 0.0), (0.0, 0.0, 1.0)), )+Myassembly.instances['Part-5-1'].faces.
findAt(((b1+b2/2, h, 0.0), (0.0, 0.0, 1.0)), )+Myassembly.instances['Part-4-1'].faces.findAt(((b1+b2+
P4h/4, h, 0.0), (0.0, 0.0, 1.0)), ((b1+b2-P4h/4, h, 0.0), (0.0, 0.0, 1.0)), ((b1+b2, h/4, 0.0), (0.0,
0.0, 1.0)), ), u1=UNSET, u2=UNSET, u3=SET, ur1=UNSET, ur2=UNSET, ur3=UNSET)
443
444 #step
445 Mymodel.BuckleStep(maxIterations=2000, name='Step-1', numEigen=350, previous='Initial', vectors=600)
446
447 #vertical load
448 if (joint == 1):
449     Mymodel.ConcentratedForce(cf2=-F, createStepName='Step-1', distributionType=UNIFORM, field='',
localCsys=None, name='VERT_F', region=Region(referencePoints=(Myassembly.referencePoints[32],
Myassembly.referencePoints[33], ))
450 Mymodel.ConcentratedForce(cf2=-Q, createStepName='Step-1', distributionType=UNIFORM, field='',
localCsys=None, name='VERT_Q', region=Region(referencePoints=(Myassembly.referencePoints[34], ))
451 elif (joint == 2):
452     Mymodel.ConcentratedForce(cf2=-F, createStepName='Step-1', distributionType=UNIFORM, field='',
localCsys=None, name='VERT_F', region=Region(referencePoints=(Myassembly.referencePoints[36],
Myassembly.referencePoints[37], ))
453 Mymodel.ConcentratedForce(cf2=-Q, createStepName='Step-1', distributionType=UNIFORM, field='',
localCsys=None, name='VERT_Q', region=Region(referencePoints=(Myassembly.referencePoints[38], ))
454
455 #mesh
456 #part 1
457 #y-direction
458 #top
459 Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/2, h,
0.0), ), ((-P1h/2, h, P1b/2), ), ((P1h/2, h, 0.0), ), ((-P1h/2, h, -P1b/2), ), ((P1h/2, h, P1b/2),
), ((P1h/2, h, -P1b/2), ), ), size=P2sy)
460 #bottom
461 Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/2, h/
4, 0.0), ), ((-P1h/2, h/4, -P1b/2), ), ((P1h/2, h/4, 0.0), ), ((P1h/2, h/4, -P1b/2), ), ((P1h/2, h/4,
P1b/2), ), ((-P1h/2, h/4, P1b/2), ), ), size=P1sy)
462 #x-direction
463 Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/4, h-
P2h/2, 0.0), ), ((P1h/4, h+P2h/2, 0.0), ), ((-P1h/4, 0.0, 0.0), ), ((P1h/4, 0.0, 0.0), ), ), size=
P1sx)
464 #z-direction
465 Mypart1.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart1.edges.findAt(((-P1h/2, h-
P2h/2, -P1b/4), ), ((-P1h/2, 0.0, -P1b/4), ), ((-P1h/2, h-P2h/2, P1b/4), ), ((-P1h/2, h+P2h/2, P1b/4
), ), ((P1h/2, h-P2h/2, -P1b/4), ), ((P1h/2, 0.0, -P1b/4), ), ((P1h/2, h-P2h/2, P1b/4), ), ((P1h/2,
0.0, P1b/4), ), ((-P1h/2, 0.0, P1b/4), ), ((-P1h/2, h+P2h/2, -P1b/4), ), ((P1h/2, h+P2h/2, P1b/4), ),
((P1h/2, h+P2h/2, -P1b/4), ), ), size=P1sz)
466 #mesh settings
467 Mypart1.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart1.faces.getByBoundingBox(-P1h/2, 0.0,
-P1b/2, P1h/2, h+P2h/2, P1b/2), technique=FREE)
468 Mypart1.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=OFF
), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart1.faces.getByBoundingBox(-P1h/2, 0.0,
-P1b/2, P1h/2, h+P2h/2, P1b/2), ))
469 Mypart1.generateMesh()
470 #part 2
471 #y-direction
472 Mypart2.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart2.edges.findAt(((b1-P1h/2,
h, 0.0), ), ((P1h/2, h, 0.0), ), ), size=P2sy)
473 #x-direction
474 Mypart2.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart2.edges.findAt(((b1/4, h+
P2h/2, 0.0), ), ((b1/4, h+P2h/2, P2b/2), ), ((b1/4, h+P2h/2, -P2b/2), ), ((b1/4, h-P2h/2, 0.0), ), ((

```

```

475     b1/4, h-P2h/2, P2b/2), ), ((b1/4, h-P2h/2, -P2b/2), ), ), size=P2sx)
476     #z-direction
477 Mypart2.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart2.edges.findAt(((b1-P1h/2,
478 h+P2h/2, P2b/4), ), ((P1h/2, h+P2h/2, P2b/4), ), ((P1h/2, h+P2h/2, -P2b/4), ), ((b1-P1h/2, h+P2h/2, -
479 P2b/4), ), ((b1-P1h/2, h-P2h/2, P2b/4), ), ((P1h/2, h-P2h/2, P2b/4), ), ((P1h/2, h-P2h/2, -P2b/4), ),
480 ((b1-P1h/2, h-P2h/2, -P2b/4), ), ((b1-P1h/2, h+P2h/2, P2b/8), ), ((P1h/2, h+P2h/2, P2b/8), ), ((P1h/
481 2, h+P2h/2, -P2b/8), ), ((b1-P1h/2, h+P2h/2, -P2b/8), ), ((b1-P1h/2, h-P2h/2, P2b/8), ), ((P1h/2, h-
482 P2h/2, P2b/8), ), ((P1h/2, h-P2h/2, -P2b/8), ), ((b1-P1h/2, h-P2h/2, -P2b/8), ), ), ), size=P2sz)
483 #mesh settings
484 Mypart2.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart2.faces.getByBoundingBox(P1h/2, h-P2h
485 /2, -P2b/2, b1-P1h/2, h+P2h/2, P2b/2), technique=FREE)
486 Mypart2.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=OFF
487 ), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart2.faces.getByBoundingBox(P1h/2, h-P2h
488 /2, -P2b/2, b1-P1h/2, h+P2h/2, P2b/2), ))
489 Mypart2.generateMesh()
490
491 #part 3a
492 #y-direction
493 #x-direction
494 Mypart3a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3a.edges.findAt(((P1h/4,
495 0.0, P1b), ), ((P1h/4, 0.0, 0.0), ), ((P1h/4, 0.0, P1b/2), ), ), ), minSizeFactor=0.1, size=P3sx)
496 #z-direction
497 Mypart3a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3a.edges.findAt(((0.0, 0.0
498 , P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4*3), ), ), ),
499 minSizeFactor=0.1, size=P3sz)
500 #mesh settings
501 Mypart3a.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart3a.faces.getByBoundingBox(0.0, 0.0,
502 0.0, P1h, 0.0, P1b), technique=FREE)
503 Mypart3a.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=
504 OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart3a.faces.getByBoundingBox(0.0, 0.0
505 , 0.0, P1h, 0.0, P1b), ))
506 Mypart3a.generateMesh()
507
508 #part 3b
509 #y-direction
510 #x-direction
511 Mypart3b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3b.edges.findAt(((P1h/4,
512 0.0, P1b), ), ((P1h/4, 0.0, 0.0), ), ((P1h/4, 0.0, P1b/2), ), ), ), minSizeFactor=0.1, size=P3sx)
513 #z-direction
514 Mypart3b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart3b.edges.findAt(((0.0, 0.0
515 , P1b/4), ), ((0.0, 0.0, P1b/4*3), ), ((P1h, 0.0, P1b/4), ), ((P1h, 0.0, P1b/4*3), ), ), ),
516 minSizeFactor=0.1, size=P3sz)
517 #mesh settings
518 Mypart3b.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart3b.faces.getByBoundingBox(0.0, 0.0,
519 0.0, P1h, 0.0, P1b), technique=FREE)
520 Mypart3b.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=
521 OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart3b.faces.getByBoundingBox(0.0, 0.0
522 , 0.0, P1h, 0.0, P1b), ))
523 Mypart3b.generateMesh()
524
525 #part 4
526 #y-direction
527 #top
528 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
529 2, h, 0.0), ), ((b1+b2-P4h/2, h, P4b/2), ), ((b1+b2+P4h/2, h, 0.0), ), ((b1+b2-P4h/2, h, -P4b/2), ),
530 ((b1+b2+P4h/2, h, P4b/2), ), ((b1+b2+P4h/2, h, -P4b/2), ), ), ), size=P5sy)
531 #bottom
532 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
533 2, h/4, 0.0), ), ((b1+b2-P4h/2, h/4, -P4b/2), ), ((b1+b2+P4h/2, h/4, 0.0), ), ((b1+b2+P4h/2, h/4, -
534 P4b/2), ), ((b1+b2+P4h/2, h/4, P4b/2), ), ((b1+b2-P4h/2, h/4, P4b/2), ), ), ), size=P4sy)
535 #x-direction
536 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
537 4, h-P5h/2, 0.0), ), ((b1+b2+P4h/4, h+P5h/2, 0.0), ), ((b1+b2-P4h/4, 0.0, 0.0), ), ((b1+b2+P4h/4, 0.0
538 , 0.0), ), ), ), size=P4sx)
539 #z-direction
540 Mypart4.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart4.edges.findAt(((b1+b2-P4h/
541 2, h-P5h/2, -P4b/4), ), ((b1+b2-P4h/2, 0.0, -P4b/4), ), ((b1+b2-P4h/2, h-P5h/2, P4b/4), ), ((b1+b2-
542 P4h/2, h+P5h/2, P4b/4), ), ((b1+b2+P4h/2, h-P5h/2, -P4b/4), ), ((b1+b2+P4h/2, 0.0, -P4b/4), ), ((b1+
543 b2+P4h/2, h-P5h/2, P4b/4), ), ((b1+b2+P4h/2, 0.0, P4b/4), ), ((b1+b2-P4h/2, 0.0, P4b/4), ), ((b1+b2-
544 P4h/2, h+P5h/2, -P4b/4), ), ((b1+b2+P4h/2, h+P5h/2, P4b/4), ), ((b1+b2+P4h/2, h+P5h/2, -P4b/4), ), ), ),
545 size=P4sz)
546 #mesh settings
547 Mypart4.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart4.faces.getByBoundingBox(b1+b2-P4h/2,
548 0.0, -P4b/2, b1+b2+P4h/2, h+P5h/2, P4b/2), technique=FREE)
549 Mypart4.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy=OFF
550 ), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart4.faces.getByBoundingBox(b1+b2-P4h/2,
551 0.0, -P4b/2, b1+b2+P4h/2, h+P5h/2, P4b/2), ))
552 Mypart4.generateMesh()
553
554 #part 5
555 if (joint ==1):
556     #y-direction
557     Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
558 P4h/2, h+P5h/4, 0.0), ), ((b1+b2-P4h/2, h-P5h/4, 0.0), ), ((b1+P1h/2, h+P5h/4, 0.0), ), ((b1+P1h/2

```

```

    , h-P5h/4, 0.0), ), ), size=P5sy)
525     #x-direction
526 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2/
4, h+P5h/2, 0.0), ), ((b1+b2/4, h+P5h/2, P5b/2), ), ((b1+b2/4, h+P5h/2, -P5b/2), ), ((b1+b2/4, h-
P5h/2, 0.0), ), ((b1+b2/4, h-P5h/2, P5b/2), ), ((b1+b2/4, h-P5h/2, -P5b/2), ), ), ), size=P5sx)
527     #z-direction
528 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, -P5b/4), ), ((b1+
b2-P4h/2, h+P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, P5b/4), ), ((b1+P1h/2, h-P5h/2, P5b/4), ),
((b1+P1h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h+P5h/2, P5b
/8), ), ((b1+P1h/2, h+P5h/2, P5b/8), ), ((b1+P1h/2, h+P5h/2, -P5b/8), ), ((b1+b2-P4h/2, h+P5h/2,
-P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/
2, -P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/8), ), ), ), size=P5sz)
529     #mesh settings
530 Mypart5.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart5.faces.getByBoundingBox(b1+P1h/2
, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), technique=FREE)
531 Mypart5.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy
=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart5.faces.getByBoundingBox(b1+
P1h/2, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), ))
532 Mypart5.generateMesh()
533
534 if (joint ==2):
535     #y-direction
536 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h, 0.0), ), ((b1+P1h/2, h+P5h/4, 0.0), ), ((b1+P1h/2, h-P5h/4, 0.0), ), ), ), size=P5sy)
537     #x-direction
538 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2/
4, h+P5h/2, 0.0), ), ((b1+b2/4, h+P5h/2, P5b/2), ), ((b1+b2/4, h+P5h/2, -P5b/2), ), ((b1+b2/4, h-
P5h/2, 0.0), ), ((b1+b2/4, h-P5h/2, P5b/2), ), ((b1+b2/4, h-P5h/2, -P5b/2), ), ), ), size=P5sx)
539     #z-direction
540 Mypart5.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart5.edges.findAt(((b1+b2-
P4h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, P5b/4), ), ((b1+P1h/2, h+P5h/2, -P5b/4), ), ((b1+
b2-P4h/2, h+P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, P5b/4), ), ((b1+P1h/2, h-P5h/2, P5b/4), ),
((b1+P1h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/4), ), ((b1+b2-P4h/2, h+P5h/2, P5b
/8), ), ((b1+P1h/2, h+P5h/2, P5b/8), ), ((b1+P1h/2, h+P5h/2, -P5b/8), ), ((b1+b2-P4h/2, h+P5h/2,
-P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/2, P5b/8), ), ((b1+P1h/2, h-P5h/
2, -P5b/8), ), ((b1+b2-P4h/2, h-P5h/2, -P5b/8), ), ), ), size=P5sz)
541     #mesh settings
542 Mypart5.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart5.faces.getByBoundingBox(b1+P1h/2
, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), technique=FREE)
543 Mypart5.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD, secondOrderAccuracy
=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart5.faces.getByBoundingBox(b1+
P1h/2, h-P5h/2, -P5b/2, b1+b2-P4h/2, h+P5h/2, P5b/2), ))
544 Mypart5.generateMesh()
545
546 if (joint ==1):
547     #part 6a
548     #y-direction
549
550     #x-direction
551 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((P4h/
4, 0.0, P4b), ), ((P4h/4, 0.0, 0.0), ), ((P4h/4, 0.0, P4b/2), ), ), ), size=P6sx)
552     #z-direction
553 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((0.0,
0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4*3), ), ), ), size
=P6sz)
554     #mesh settings
555 Mypart6a.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart6a.faces.getByBoundingBox(0.0,
0.0, 0.0, P4h, 0.0, P4b), technique=FREE)
556 Mypart6a.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD,
secondOrderAccuracy=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart6a.faces.
getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b), ))
557 Mypart6a.generateMesh()
558 elif (joint ==2):
559     #part 6a
560     #y-direction
561
562     #x-direction
563 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((P4h/
4, 0.0, P4b), ), ((P4h/4, 0.0, 0.0), ), ((P4h/4, 0.0, P4b/2), ), ), ), size=P6sx)
564     #z-direction
565 Mypart6a.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6a.edges.findAt(((0.0,
0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4*3), ), ), ), size
=P6sz)
566     #mesh settings
567 Mypart6a.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart6a.faces.getByBoundingBox(0.0,
0.0, 0.0, P4h, 0.0, P4b), technique=FREE)
568 Mypart6a.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD,
secondOrderAccuracy=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart6a.faces.
getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b), ))
569 Mypart6a.generateMesh()
570
571     #part 6b
572     #y-direction
573
574     #x-direction

```

```
575 Mypart6b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6b.edges.findAt(((P4h/
576 4, 0.0, P4b), ), ((P4h/4, 0.0, 0.0), ), ((P4h/4, 0.0, P4b/2), ), ), size=P6sx)
577 #z-direction
578 Mypart6b.seedEdgeBySize(constraint=FINER, deviationFactor=0.1, edges=Mypart6b.edges.findAt(((0.0,
579 0.0, P4b/4), ), ((0.0, 0.0, P4b/4*3), ), ((P4h, 0.0, P4b/4), ), ((P4h, 0.0, P4b/4*3), ), ), size
580 =P6sz)
581 #mesh settings
582 Mypart6b.setMeshControls(elemShape=QUAD_DOMINATED, regions=Mypart6b.faces.getByBoundingBox(0.0,
583 0.0, 0.0, P4h, 0.0, P4b), technique=FREE)
584 Mypart6b.setElementType(elemTypes=(ElemType(elemCode=S4, elemLibrary=STANDARD,
585 secondOrderAccuracy=OFF), ElemType(elemCode=S3, elemLibrary=STANDARD)), regions=(Mypart6b.faces.
586 getByBoundingBox(0.0, 0.0, 0.0, P4h, 0.0, P4b), ))
587 Mypart6b.generateMesh()
588 #imperfection
589 Mymodel.keywordBlock.synchVersions(storeNodesAndElements=False)
590 Mymodel.keywordBlock.replace(304, '\n*Output, field, variable=PRESELECT\n*NODE FILE\nU')
591 #job
592 mdb.Job(atTime=None, contactPrint=OFF, description='', echoPrint=OFF, explicitPrecision=SINGLE,
593 getMemoryFromAnalysis=True, historyPrint=OFF, memory=90, memoryUnits=PERCENTAGE, model='LBA-frame-'+
594 id, modelPrint=OFF, multiprocessingMode=DEFAULT, name='LBA-job-'+id, nodalOutputPrecision=SINGLE,
595 numCpus=1, numGPUs=0, queue=None, resultsFormat=ODB, scratch='', type=ANALYSIS, userSubroutine='',
596 waitHours=0, waitMinutes=0)
597
598
599
600
601
602
603
604
605
606
607
608
609
610
611
612
613
614
615
616
617
618
619
620
621
622
623
624
625
626
627
628
629
630
631
632
633
634
635
636
637
638
639
640
641
642
643
644
645
646
647
648
649
650
651
652
653
654
655
656
657
658
659
660
661
662
663
664
665
666
667
668
669
670
671
672
673
674
675
676
677
678
679
680
681
682
683
684
685
686
687
688
689
690
691
692
693
694
695
696
697
698
699
700
701
702
703
704
705
706
707
708
709
710
711
712
713
714
715
716
717
718
719
720
721
722
723
724
725
726
727
728
729
730
731
732
733
734
735
736
737
738
739
740
741
742
743
744
745
746
747
748
749
750
751
752
753
754
755
756
757
758
759
760
761
762
763
764
765
766
767
768
769
770
771
772
773
774
775
776
777
778
779
780
781
782
783
784
785
786
787
788
789
790
791
792
793
794
795
796
797
798
799
800
801
802
803
804
805
806
807
808
809
810
811
812
813
814
815
816
817
818
819
820
821
822
823
824
825
826
827
828
829
830
831
832
833
834
835
836
837
838
839
840
841
842
843
844
845
846
847
848
849
850
851
852
853
854
855
856
857
858
859
860
861
862
863
864
865
866
867
868
869
870
871
872
873
874
875
876
877
878
879
880
881
882
883
884
885
886
887
888
889
890
891
892
893
894
895
896
897
898
899
900
901
902
903
904
905
906
907
908
909
910
911
912
913
914
915
916
917
918
919
920
921
922
923
924
925
926
927
928
929
930
931
932
933
934
935
936
937
938
939
940
941
942
943
944
945
946
947
948
949
950
951
952
953
954
955
956
957
958
959
960
961
962
963
964
965
966
967
968
969
970
971
972
973
974
975
976
977
978
979
980
981
982
983
984
985
986
987
988
989
990
991
992
993
994
995
996
997
998
999
```