

Development test report airbus swivel valve

Citation for published version (APA):

Bruls, H. J. (1991). Development test report airbus swivel valve. (DCT rapporten; Vol. 1991.084). Technische Universiteit Eindhoven.

Document status and date: Published: 01/01/1991

Document Version:

Publisher's PDF, also known as Version of Record (includes final page, issue and volume numbers)

Please check the document version of this publication:

• A submitted manuscript is the version of the article upon submission and before peer-review. There can be important differences between the submitted version and the official published version of record. People interested in the research are advised to contact the author for the final version of the publication, or visit the DOI to the publisher's website.

• The final author version and the galley proof are versions of the publication after peer review.

• The final published version features the final layout of the paper including the volume, issue and page numbers.

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DEVELOPMENT TEST REPORT AIRBUS SWIVEL VALVE

Report prepared by: H. Bruls

Test performed by:P. v.d. MortelP. v. BerloH. BrulsTest supervised by:R. Pander MaatR & DT. SijpkesR & DW. v. GogProc./Manuf.Technol.H. v. AssenProjekt/Kap.BeheerH. BrouwerProcurement group

Rapport WFW 91.084 Report 2199-59000/91-425HB

April-August 1991

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SUMMARY

New airbus 330/340 aeroplanes will probably first fly in Oktober 1991. DAF special products has engineering reponsibility of the steering actuators and Swivel Valves. The Swivel Valve is a hydraulic valve providing reversal of hydraulic pressure during steering operations.

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A number of problems arised about its operation during production, assembling and acceptance testing :

- chipping of chrome from pocket edges
- completely blocking during rotation without pressure
- appearance of deep scratches during acceptance testing

Although up till now, the Swivel Valves have performed without any problems during qualification tests at the client's, DAF wants to test them to ensure the proper functioning during its complete endurance life. Alternative Swivel Valve configurations are also tested to have a fall back solution when the basic configuration fails. During the test a number of measurements are done, to build up knowledge about the mechanism that is involved in wear.

Conclusions after 50,000 flights endurance testing :
All Swivel Valve configurations satisfied concerning wear (the ionic nitrided configuration most).

- All Swivel Valve configurations showed leakage.
- None of the seals used in the valves operated satisfactory.

Recommendations :

- The supply of basic Swivel Valve configurations can continue.
- The seal problem has to be solved.
- The ionic nitride treatment has to be investigated further.

1 INTRODUCTION

The Airbus 330/340 aeroplanes will first fly in October 1991. The nose landing gear of such an aeroplane consists of two actuators to provide steering during ground manoeuvring. A Swivel Valve, which is a hydraulic valve, is mounted on these actuators and provides a fluid direction change within them during rotation of the nose landing gear.

DAF Special Products has engineering responsibility of the steering actuators and the Swivel Valves of the Airbus 330/340. During mounting of several Swivel Valves and rotating just for a few times without pressure, a few blocked completely, some showed several deep marks of seal grooves and pocket edges on contact surfaces, especially in the house and chrome chipped from pocket edges of other. Hence there is anxiety about current Swivel Valve configuration proper functioning during its endurance life. Because of these doubts, it was decided to perform an endurance test and to test Swivel Valves on endurance. Not only the Swivel Valve standard configuration but also alternative configurations are being tested. The purpose of this test is to get certainty about the configuration, that will satisfy not only functional requirements, but also will perform over its lifetime. Further important aspects for executing this test are:

- getting a better view of factors that are involved in wear
- getting a better view of mechanism that is involved
- building up own general knowledge
- coming up with solutions in case the Swivel Valve doesn't survive the endurance tests executed at Messier Buggatti, a French landing gear manufacturer and designer of the Airbus 330/340 nose landing gear, and principal contractor.

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2 Problem Definition

2.1 The Swivel Valve

The Swivel Valve is mounted on both steering actuators of the nose landing gear and provides reversal in flow direction of the hydraulic fluid used in these actuators. In this way the steering actuator changes from push to pull or visa versa. The Swivel Valve consists of two parts, a house and a core (see figure 2.1.1), that rotate in relation to each other. Further detailed information about the Swivel Valve operation are shown in enclosure A.



Figure 2.1.1 : Drawing of the Swivel Valve mounted on the actuator.

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2.2 Configuration Development of the Swivel Valve

During the design of the Swivel Valve, the basic material used for core and house was changed from a basic carbon-steel (4340) to stainless steel. The first design did not provide any surface treatment for the contact area between core and house. This caused galling of the assembly during acceptance testing. Therefore the core of the Swivel Valve was partially chrome-plated. The not chrome-plated area over the seal groove was reduced in diameter to prevent galling in this area.

At first the chrome-plating seemed to solve the problem, but a number of valves still blocked during acceptance testing.

During the design, a special seal for rotary applications was proposed by DAF, but the principal contractor Messier disagreed on this selection and ordered o-seals to be used. These are not preferred for rotary applications, but the selection was made on the experience with simular valves and from the standpoint of economy and logistics.

2.3 Current Swivel Valve Configuration

The current material of the Swivel Valve house is Z15CN17-03 without further treatment and the core is made out of 15-5PH (see table 2.3.1 for chemical structure).

The surface of the core is chrome plated till 1.5 mm from the seal groove and it has a nominal thickness of 100 um (see figure 2.3.1).

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	house	core
material	Z15CN17-03	15-5PH H1025
c	0.12-0.20	< 0.05
Si	< 1.00	0.05-0.60
Mn	< 1.00	< 0.6
Р	< 0.025	< 0.03
s	< 0.025	< 0.03
Cr	15.0-18.0	14.5-15.5
Ni	2.0-3.0	4.2-5.0
Cu	-	2.8-3.5
Nb+T	-	0.15-0.30
N	-	< 0.05
	I	
tensile strength N/mm ²	880-1080	1125-1320
hardness HB	262-326	
hardness HRC		35-42
hardness HV	275-340	350-410

Table 2.3.1 : Chemical structure of house and core.



Figure 2.3.1 : The current core configuration with a harchrome surface.

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The manufacturing method for the surface treatment of the core is : there is put about 250 um chrome on the surface of the core and then the pocket (4 rectangle shaped pits within the running surface of the core) edges are grinded. After this the diameter of the core is grinded cylindric to about 100 um . Finally the core is matched on the house with a nominal play of 8 to 12 um on the diameter.

2.4 Problem Diagnose

The actual problems that remained after the design had been progressed to this stage (as described in section 2.3) are listed below.

- A number of valves still blocked during non-pressure turning at acceptance testing.
- At the sharp edges of the pockets, that are crucial to the switching characteristics of the Swivel Valve, chrome chips break away during final grinding. This causes a quality problem for production and can cause critical failure if chips are caught between core and house.
- Deep scratches appear in the house and on the core after non-pressure rotating of the assembly.

Because of these problems there was concern, whether the Swivel Valve could operate properly over its service life. Therefore it was decided that an endurance test would be performed, including a number of alternative designs to develop an optimal configuration for the Swivel Valve.

2.5 REM-analysis of a Swivel Valve Core

One core which blocked completely during acceptance testing due to binding and which had scratches on the outside has been analysed by DAF's Cental Laboratory. The core has been analysed with a REM for visual damage at Philips PMF-CMTI.

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2.5.1 Results of the analysis

The running surface of the core has been analysed. It shows a number of deep scratches in rotation direction and chipped chrome particles at the edge of the pockets.

Figure 2.5.1.1 shows a macrophotograph of the chipped chrome particles in the left upper edge of one of the two pockets, which is nearest to the seal groove of the core. In this same pocket chipping of the chrome is also found at its right side.

The side of the seal groove and near the pocket shows a rather deep scratch in rotation direction. The scratches in axial direction are probably the result of demounting of the core and the house of the Swivel Valve.

The figure shows that the chrome stops somewhere between the pocket edge and the seal groove and not 1.5 mm from the seal groove as specified in the drawing of the core.

2



figure 2.5.1.1 : Macrophotograph of chipping particles at the upper left edge of one of the two pockets nearest to the seal groove.

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Figure 2.5.1.2 shows a REM-photograph of the edge of the pocket shown in figure 2.5.1.1 with the chipping chrome. It shows scratches in the chrome surface and chrome in the pockets. The chipping has taken place in the chrome surface and not in the base material. This has been verified by a EDX-analysation (a qualitative analysation of the chemical composition). This means that the firmness between the chrome and the base material has been good.



Figure 2.5.1.2 : REM-photograph of the edge of the pocket.

Figure 2.5.1.3 shows a REM-photograph near figure 2.5.1.2 at the running surface. In the lower right corner the chipping chrome can be seen. The pollution of the chrome in axial direction contains Fe and is probably from the damaged house.

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Figure 2.5.1.3 : REM-photograph near figure 2.5.1.2 on the running surface.

Figure 2.5.1.4 shows a REM-photograph which is a detail of figure 2.5.1.3 just above the chipped edge of the pocket. It shows hair cracks in the chrome surface.



Figure 2.5.1.4 : REM-photograph above the chipped edge of the pocket.

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Figure 2.5.1.5 shows a detail of the deep scatch shown in figure 2.5.1.1. Left and right of the groove Fe can be found on the chrome surface probably because of material transfer from the house to the core. The groove itself contains chrome, so the damage has not been penetrated through the chrome surface.



Figure 2.5.1.5 : Detail of the deep scratch in figure 2.5.1.5.

Figure 2.5.1.6 shows a macrophotograph of a groove where grating has taken place. This groove is located at the crossing chrome/base material at the seal groove side diametrical of the pocket from figure 2.5.1.6.



Figure 2.5.1.6 : Macrophotograph of a groove consequent on grating.

Figure 2.5.1.7 shows a detail photograph.



Figure 2.5.1.7 : Detail photograph of the groove consequent on grating.

On the other side of the running surface of the core, in axial direction, 2 grooves are located as high as the seal groove of the house (figure 2.5.1.8).



Figure 2.5.1.8 : The other side of the running surface of the core, in axial direction.

The grooves in the crome in rotation direction are probably caused by hard particles, which are embedded in the rather soft material of the house and then cutted in the core. Because of the wear these particles have caused in the core the house is damaged and material transfer from house to core has taken place. Because the house wasn't analysed Central Laboratory couldn't tell where the particles came from, but they are probably chrome particles or corund abrasive particles. Remarkable are also the grooves as high as the seal groove of the house.

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Chipping of the chrome from one pocket is caused by violence, secondary cracks are also found. It is not sure whether the crome is chipped during grinding, testing or dissambling. The chrome is not chipped from the base material, but in the chrome surface itself. This means that the firmness of the chrome with the base material is good. Because of the thick chrome (250 um) during grinding to a size of 100 um, chipping takes place. Because of the extra surface build up at the edges the surface will be there probably between 300 and 400 um thick.

At the crossing chrome/base material at the side of the seal groove of the core grating has taken place. The cause is probably the running out of the chrome. The axial scratches are probably caused by demounting.

2.5.3 Recommendations

- Use a chrome thickness of 50-75 um and grinding as little as possible.
- Use a chrome surface on the complete running surface.
- Round the edges of the pockets, so less thickness
- build up of the chrome.

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- Keep your hydraulic system as clean as possible.

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3 TESTCONFIGURATIONS

One problem concerning the standard configuration of the Swivel Valves is possible chipping of chrome from the pocket edges during rotation. If these chrome particles are big enough, they can cause blocking between the core and the house.

Because the test for the Swivel Valve starts in a rather late stage (some Swivel Valves have already been send to Messier), the changes concerning the Swivel Valve can not be drastic. That is why only minor modifications can be allowed for the Swivel Valve design and the form, fit and function has to remain unchanged.

The standard configuration for the Swivel Valve can be changed in the following way : - changes in surface treatment

- geometric changes
- using other basic material

3.1 Surface Treatment

The Swivel Valve can be changed in choosing another surface treatment.

The treatments that have been investigated are listed below :

- Electroless Nickel Plating
- Hardening
- Induction Hardening
- Reduction
- Nitriding
- Ionic Nitriding
- Crome Plating
- Hard Anodizing
- Chemical Vapour Deposition

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- Physical Vapour Deposition
- Ceramic
- Plasma

These treatments have been di Laboratory, Plating Departmen	iscussed with the Central nt and Production Engineering					
and their recommendations are listed below (these						
recommendations refer to surf	face treating the core, because					
according to them surface tre	eating the core would be easier					
than gurfage treating the boy						
chan surface creating the not	ise) .					
- Chrome Plating	: possible if executed					
Disting	correctly					
- Electroless Nickel Plating	correctly					
- Uardoning	· product is already hardened					
- Hardening	Tust bardening is not enough					
	The that age the material					
	in that case the material					
	complication still remains					
	stainless steel versus					
Toduction Henderening	stainless steel.					
- Induction Hardenening	: same comment as hardening					
- Reduction	: same comment as hardening					
- Nitriding	: This is not possible because					
	of the needed temperature of					
	515°C. Stainless steel					
	looses its resistance to					
· · · · · · · · ·	corrosion.					
- Ionic Nitriding	: This is possible and can be					
	done by Mamesto, a company					
	in Arcen Holland. This					
	treatment will cause no lost					
	of resistance to corrosion.					
- Chromating	: This isn't usefull, because					
	of the already presents of					
	Chrome in the basic					
	material.					
- Hard Anodizing	: Only possible when using					
	Aluminum as basic material.					
- Chemical Vapour Deposition	: This isn't possible because					
	of the high working					
	temperature 850-1000°C.					

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-	Physical	Vapour	Deposition	:	This i	s possible	but not	yet
					used i	in aeronauti	lcs.	
-	Ceramic			:	The sa	me comment	as for	
					PVD/CV	7D.		
	Plasma			:	This i	s possible	with onl	Ly
					Molybd	lenum.		

Regarding this the cores during the first test will be treated as follows :

- Chrome Plating
- Ionic Nitride

and during an eventual second test :

- Physical Vapour Deposition
- Molybdenum Plasma

3.2 Geometric Changes

The geometric changes will be changes especially in the pocket size and shape on the core.

The <u>size</u> of the pockets can be decreased to create a larger bearing surface and reduce wear. To decrease the size of the pockets, their positions have to change. The position of the pockets has to be lined up, as can be seen in figure 3.2.1.

The <u>shape</u> of the pockets is also important regarding wear aspects.

A large radius on the pocket corners reduces the possibility of cutting of the pockets in the house (see figure 3.2.2).



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Figure 3.2.2 : Increasing radius of pocket corners.

The limitation for increasing the radius of the pockets edges is the switching moment of the Swivel Valve. Therefore the pocket edges must be as sharp as possible. The limitation in decreasing the bore holes within the pockets, is the pressure drop over the Swivel Valve, which may not be too large.

Regarding this the Swivel valve core will be changed for the first test round in :

- increasing radius of pocket corners

- decreasing bore holes within pockets

3.3 Material Changes

Besides the stainless steels as selected for the current Swivel Valve configuration it is allowed to change back to previously envisaged carbon-steel. However it must be guaranteed that parts will be properly protected against corrosion.

The corrosion protection is a specified demand from Messier, because of earlier corrosion problems on simular valves.

For the first test round it was decided to stay with the basic material as selected for the current design.

3.4 Seal Change

The use of an o-seal in rotary applications is not the most preferred solution. Therefore it was expected that the o-seal would fail during the test. To provide a backup solution, an alternative seal was selected, specially designed for rotary applications.

From the seal manufacturer Geene Tweed, the ENER-CAP seal was selected.

From the catalogue, the glas-filled cap was selected for chrome plated surfaces, but the manufacturer recommended carbon filled caps for our applications.

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The 4 choosen testconfigurations of the Swivel Valve for the first test round are :

type 1	l) Basic Design	Standard unit
type 2	2) Improved Design	- Pocket corner with radius 3mm
	Chrome	- Completely chromed core
		- Reduced chrome layer (50um)
type 3	3) Improved Geometry	- Pockets in line, cross
	Chrome	drillings under angle
		- Pocket corner with radius 3mm
		- Completely chromed core
		- Reduced chrome layer (50um)
		- Alternative seal, Greene
		Tweed type : enercap
type 4	4) Improved Geometry	- Pockets in line, cross
	Ionic Nitriding	drilling under angle
		- Pocket corners 4mm radius
		- flange flash-chromed
		- Completely ionic nitrided
		core
		- Alternative seal, Greene

Tweed type : enercap

The drawings of the cores used for the endurance test are shown in enclosure II drawingno : - SP10288

- SP10289

- SP10290
- SP10291

The pocket edges and seal groove of the test configurations are not completely representative for the production units, because the order of machining of the outside diameter and the pockets was reversed. First the pockets should have been grinded to size, to allow small chips on the edge to

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be machined away during final matching of the outside diameter.

A lot of attention has been put into the breaking of the pocket edges, done under a microscope with a lapping stone. This will remain a critical operation during series production, crucial to the characteristics of the valve.

The requirement for chroming of cores type 2 and 3 were a maximum chrome layer of 75um. After chroming, the minimum layer thickness was 60um and the maximum around 100um. Chrome plating was done up to the edge of the pocket. The final grinding sequence was first the pockets and then the outer diameter. This caused chipping of the chrome, because of the fact that both the chrome and the base material were grinded. Therefore it was decided to machine the pockets slightly oversize and allow the core to be chromed over the pocket edge.

After stripping and chrome plating, the grinding sequence aws reversed by accident. Therefore, small chrome chips that broke off during grinding could not be grinded away during final matching. Therefore the quality of the pocket edges is not completely representative for normal production.

Because the cores of type 2 and 3 are chromed over the seal groove, the seal groove has to be grinded after chroming. After examination of the cores after the testing, it appeared that the final grinding of the seal groove had not been done.

The core of type 4 has been ionic nitrided by Mamesta in Lomm. After return to DAF the core is black on the running surface and blue everywhere else. A dummy core which has also been ionic nitrided has been analysed by Central Laboratory. The hardness of the pocket edges and in the pockets is 1100 HV and the hardness of the running surface is 400 HV, which is the same hardness as the base material of 15-5 PH. The bearing surface of the core has no nitride surface at all. The ionic nitride process has failt. Measuring the diameter of the bearing surface of the core before ionic nitriding results in 37.877 mm and after ionic nitriding results in 37.871 mm and so it appeared to have shrunck 6 um. The temperature of this ionic process was just below 500°C.

The same core of type 4 has been nitrided for a second time at a higher temperature (570°C), but this temperature is above the PH-temperature of the material (550°C). Because of the advice of Mamesta Central Laboratory to ionic nitride the core better a higher temperature is used. A dummy core which has also been ionic nitrided has again been analysed by the Central Laboratory. The hardness on the core is about 1100 HV on the whole bearing surface including the pockets, which is satisfying. After measuring the core, its cylindricity is not satifying (see table 3.5.1).

C	ylindricity	of	the	ionic	nitrided	core (mm)	
 	measured		al	llowed	đi	ifference	
 	0.0204		(0.0050		0.0154	

Table 3.5.1 : Cylindricity of the bearing surface of core type 4 before testing.

The core again has shrunk to a diameter of 37.842 mm. Because of the shrunk core a new house is matched on the diameter of the core.

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The lifetime of an Airbus 330/340 aeroplane is 50,000 flights. Every 20,000 flights the landing gear is completely overhauled. During this overhaul a Swivel Valve which fails to meet the requirements can be replaced by a new one. Therefore the operational life of a Swivel Valve is required to be 50,000 flights with an absolute safety minimum of 20,000 flights.

4.1 Nose Landing Gear Endurance Spectrum

- The endurance spectrum of the nose landing is as follows :
- a) Manoeuvre A

Controlled Manoeuvre

rotation	:	± 50°
rotation speed	:	0.145 rad/s
frequency	:	2 per flight

b) Manoeuvre B

Oscillating Manoeuvre

rotation	:	± 3°
rotation speed	:	0.209 rad/s
frequency	:	15 per flight

And all these manoeuvres are carried out with 3 different pressures, namely :

1)	20.6	MPa
2)	14.1	MPa
3)	6.7	MPa

These pressures are in the proportion of (1) : (2) : (3) = 1 : 99 : 900

4.2 Relation between Nose Landing Gear and Swivel Valve

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The schematic mechanism between nose landing gear and Swivel Valve is shown in figure 4.2.1.

AC rotates around A and represents the rotation of the nose landing gear. BC rotates around B and represents the rotation of the actuator and so also of the Swivel Valve which is mounted on the actuator. At point C the actuator is attached to the nose landing gear. BC is variable in length.

ðse landing gear actuator swivel valve

Figure 4.2.1 : Schematic mechanism of the nose landing gear.

The Swivel Valve rotation versus nose landing gear rotation during manoeuvre A and B are shown in figure 4.2.2 and 4.2.3.







Figure 4.2.3 : Swivel Valve rotation versus nose landing gear rotation during manoeuvre B.

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The Swivel Valve angular velocity versus the nose landing gear rotation during manoeuvre A and B is shown in figure 4.2.4 and 4.2.5.



Figure 4.2.4 : Swivel Valve angular velocity versus nose landing gear rotation during manoeuvre A.



Figure 4.2.5 : Swivel Valve angular velocity versus nose landing gear rotation during manoeuvre B.

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4.3 Swivel Valve Endurance Spectrum

The Swivel Valve endurance spectrum is shown below :

- It is required that the Swivel Valve operates correctly during 50,000 flights.

From figures 4.2.2,4.2.3,4.2.4 and 4.2.5 it is clear that the Swivel Valve manoeuvres are :

a) Manoeuvre A

Controlled Manoeuvre

rotation range : -42.41° -> 43.30° maximum angular velocity : 0.33 rad/s frequency : 2 per flight

b) Manoeuvre B

Oscillating Manoeuvre rotation range : 27.33° -> 34.50° maximum angular velocity : 0.44 rad/s frequency : 15 per flight

with 3 different pressures, namely :

1) 20.6 MPa 2) 14.1 MPa 3) 6.7 MPa

in the proportion of (1) : (2) : (3) = 1 : 99 : 900

4.4 Test Programme

Every selected Swivel Valve configuration will undergo a test programme doing different manoeuvres with different pressures as specified in its endurance spectrum (see section 4.3).

The required time between overhaul of the Swivel Valve is 20,000 flights. To ensure proper functioning and to eliminate the statistical probability of having an occasional good product the test will be performed over 50,000 flights.

When we assume the valves' endurance lifes to be distributed Gaussian and the standard deviation equal to 10,000 flights, the probability for one configuration surviving its test and not surviving 20,000 flights is 0,13 per cent.

One cycle, which will be further used in this report, represents one relative harmonic rotation of the Swivel Valve.

To limit the number of different loadings within a test block, the number of cycles at the lowest pressure (6.7 MPa) are added to the number of cycles at the middle pressure (14.1 MPa). This will increase deformation of the house and hence increase wear over lifetime.

To speed up the test, higher angular velocities are selected for the specified manoeuvres of the Swivel Valve. The values choosen are guided by design constrains of the electric motor and the transmission pulleys (see enclosure IV). The values for angular velocities and rotation range during the test are compared to the one in reality and shown in table 4.4.1 and plotted in figure 4.4.1 and 4.4.2.

	test	reality
max. rotation range (deg)	-44.93°-44.99° 24.38°-38.73°	-42.41-43.30 27.33-34.50
<pre>max. angular velocity at max. rotation (rad/s) max. angular velocity</pre>	2.04	0.33
at min. rotation (rad/s)	1.17	0.44

Table 4.4.1 : Comparison between test and reality.

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Figure 4.4.2 : Comparison the angular velocities and rotations during manoeuvre B.

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The tests will be carried out in blocks with a block representing a part of the Swivel Valve life (10,000 flights). Between the blocks, measurements will be taken and judgements will be made concerning the status of the Swivel Valve.

The total number of cycles during the endurance test, covering 50,000 flights for every valve is shown in table 4.4.2.

rotation(deg)/pressure(Mpa)	20.6	14.1	total
$ -44.93 \rightarrow 44.90$ (A) $ 24.38 \rightarrow 38.73$ (B)	100 750	99,900 749,250	100,000 750,000
total	850	849,150	850,000

Table 4.4.2 : Number of cycles for the endurance test of a Swivel Valve configuration with different pressures and rotation angles.

The total number of cycles will be divided into 5 equal sized blocks (10,000 flights). The total number of cycles one Swivel Valve configuration has to make during one block is shown in table 4.4.3.

number of cylcles per block rotation(deg)/pressure(Mpa)	20.6	14.1	total
-44.93 -> 44.99 (A) 24.38 -> 38.73 (B)	20 150	19,980 149,850	20,000 150,000
total	170	169,830	170,000

Table 4.4.3 : Number of cycles for a Swivel Valve configuration during one block with different pressures and different rotation angles.

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Knowing the angular velocity of the driving disc 0.58 rotations per second for the range of -44.93° -> 44.99° and 1.5 rotations per second for the range 24.38° -> 38.73° (enclosure III), the time scheduling can be calculated and is shown in table 4.4.4.

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time in hours : rotation(deg)/pressure(Mpa)	20.6	14.1	total
-44.93 -> 44.99 24.38 -> 38.73	0.06	55.5 138.75	55.56 138.89
total	0.20	194.25	194.45

Table 4.4.4 : Time table for the test using different pressures and rotation angles.

4.5 Measurements

Measurements that will be taken of all configurations before testing :

- Torque between house and core
- Leakage (acceptance test)
- Draw sample of the fluid
- Roundness
- Cylindriness -
- Dimensional control

Measurements between the blocks and after the test :

- Torque between house and core
- Leakage (acceptance test)
- Draw sample of the fluid
- Roundness -
- Cylindriness -

Measurements after the test :

- Drop test (type 3 or 4)
- Dimensional Control

Measurements after failing the test :

- Draw sample of the fluid
- Roundness
- Diameter
- + profile measurement
- Cylindriness
- REM-analysis

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Discussing the measurements :

- Dimensional Control : This is a measurement of a few dimensions of core and house (see figure V.1, V.2 and V.3).
- Profile Measurement :

A profile measurement of the house consists of 2 straightness measurements between port A and B on both sides, 2 roundness measurements with one between the ports and the seal groove and one on the other side of the ports and one cylindricity measurement on the inside of the house. The profile measurement of the core consist of 4 staightness measurements between the 4 pockets, 2 roundness measurements with one between the seal groove and the pockets and one on the other side of the pockets.

- Surface Observation :

On an unfold the inside of the house and the outside of the core surfaces are drawn.

- Draw Sample of the Fluid :

A sample of the fluid is taken at the non-pressure side of the house to let it analyse by a laboratory. Chipped particles coming from the inside of the Swivel valve can be spotted.

- Torque Measurement

The maximum torque between house and core is measured.

- Leakage Test:

The internal leakage of the Swivel Valve will be measured for a number of different relative angles between core and house mounted. A Pressure of 206 bars is put on one port and leakage during one minute is measured at another port (see enclosure V) .

- Drop Test :

The pressure drop through the valve is measured for Swivel Valve configurations type 3 or 4 is measured, because they are changed geometric (see V.3.11).
The measurements are numbered counting from 0 to 5. Measurement 0 represents the measurements done before the endurance test starts and measurement number 5 represents the measurements done after the endurance test. Measurement numbers 1,2,3 and 4 are measurements between the blocks.

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NOTE : Because performing the leakage test and a profile measurement at the same day would take too much time, these measurements are split. Leakage will be measured during measurement 0,1,3 and 5 and profile will be measured during measurement 0,2,4 and 5.

5 TEST FRAME

To achieve the Swivel Valve to do a reciprocating rotation the following mechanism was used as shown in figure 5.1.



Figure 5.1 : Mechanism to achieve a reciprocating rotation of one Swivel Valve.

The driving disc rotates with a constant angular velocity in one direction and so the Swivel Valve rotates around and back because of its longer arm.

To save time 4 Swivel Valves are tested at the same time under the same circumstances. The driving motion is provided by an electric motor (see enclosure III for the electric motor calculation). To use the power of the electric motor optimal the Swivel Valves are positioned in a square and the driving disc is located in the middle (see drawing SP10263 enclosure II).

The 2 different rotation ranges of the Swivel Valve during testing will be achieved by using 2 arms with different length and 2 and plugging in on 2 different diameters of the driving disc. The 2 different angular velocities will be achieved by using 2 pairs of pulleys and 1 drive belt. (for angular velocity graph and rotation graphs see enclosure IV.

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6 RESULTS

See also enclosure V for test results (axis definitions in figures V.4 and V.5, port and angle definitions in figure V.6)

Dimensional control :

- The diameter clearances between core and house of all 4 types don't satisfy their drawing demands (8-12um diameter clearance) before and also after testing (see figure 6.1). - The diameter clearance for type 4 is the highest (at the end of the test 41um and at the beginning not measured, but probably also not according to drawing demands).

- The diameter clearance growth between type 1 and 2 is exactly the same (17-14=3um growth) and is larger than for type 3 (19-18=1um).



Diameter clearance between core and house type 1,2,3,4

measurement number

Figure 6.1 : Diameter clearance between core and house of type 1,2,3 and 4 during the test.

profile measurement :

- The cylindricities of all houses (figure 6.2) are are not according to drawing demands (0.002mm).

- At the end of the test, the cylindricity values are larger than at the beginning.

- The cylindricities for type 1 and 2 show a simular behavier: growth between 0 and 2, drop between 2 and 4 and enormous growth between measurement 4 and 5.

Type 4 shows a sligth cylindricity drop between measurement 4 and 5.

- The cylindricity growth for type 3 and 4 between measurement 2 and 5 (type 3 : 0.0037mm and type 4 : 0.0045mm) is less than for type 1 and 2 (type 1 : 0.0090mm and type 2 : 0.0115).



Figure 6.2 : Inside house cylindricity curve of type 1,2,3 and 4.

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Figure 6.3 shows house roundness curves (z=-17.00 mm) for all 4 types.

- The roundness of type 2 and 3 are in the beginning according to drawing demands (0.0050mm) and of type 1 nearly (0.0051mm).

- The curves of type 1 and 2 show a simular behavier : growth between 0 and 2, drop between 2 and 4 and growth between measurement 4 and 5.

- The curve of type 4 shows a remarkable drop between begin and end.



measurement number



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The inside roundness (z=-32.00mm) curves for all 4 types are shown in figure 6.4.

- Type 1 and 2 are at the beginning according to drawing demands.

- The curves of type 1 and 2 show a simular behavier : growth between 0 and 2, drop between 2 and 4 and growth between measurement 4 and 5.

- The curve of type 3 is rather stable.



Figure 6.4 : House roundness (z=-32.00mm) curves of type 1, 2, 3 and 4.

The core cylindricity curves of all 4 type are shown in figure 6.5.

- None of the cylindricities are according to drawing demands (0.005mm).

- The cylindricity of core type 4 is the highest.

- The cylindricity curve for type 4 shows an enormous growth between measurement 0 and 2 followed by an enormous drop between measurement 2 and 4.

- The cylindricity curves for core type 1 and 2 show a simular behavier : between 2 and 4 drop and between measurement 4 and 5 growth.

- The cylindricity curves for core type 3 and 4 also show a simular behavier : between 0 and 2 growth, between 2 and 4 drop and between measurement 4 and 5 sligth drop.

- The cylindricity growth between begin and end is not much and for type 3 even a sligth drop (-0.001mm).



Figure 6.5 : Core cylindricity curves of type 1,2,3 and 4.

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Figure 6.6 shows core roundness (z=-10.00mm) curves of type 1,2,3 and 4.

- Only type 2 is according to drawing demands (0.005mm) at the beginning of the test.

- Type 4 has the highest roundness.

- The last measured value is for all types equal or a

little bit less than the first measured.

- The variation is the highest for type 2.



Figure 6.6 :Core roundness (z=-10.00mm) curves of type 1,2,3 and 4.

Figure 6.7 shows core roundness (z=-34.00mm) curves of type 1,2,3 and 4.

- Only type 2 is according to drawing demands (0.005mm).

- Type 4 shows the highest roundness and is continuously dropping during the test.

- The curves of type 2 and 3 are rather stable.



measurement number

Figure 6.7 : Core roundness (z=-34.00mm) curves of type 1,2,3 and 4.

Surface observation :

- Before the test starts core type 1,2 and 3 shows a few little chipped chrome particles at the pocket metering edges.

- During the test nearly no chrome particles have been chipped from the pocket edges of type 1,2 and 3.

- Type 1,2 and 3 show marks of seal grooves in house and core.

- House of type 4 shows mark of seal groove.

- Core of type 4 shows less damage at the surface of all 4 types.

- House of type 4 shows a few, but rather deep stcatches in vertical direction.

- All 4 houses show a black mark within the seal groove mark.

Leakage test :

Figures 6.8,6.9,1.10,6.11 and 6.12 shows the internal leakage of type 1,2,3 and 4.

- Type 4 shows the highest leakage.

- If the leakage is high, then the leakage mostly increases. If the leakage is not high sometimes the leakage decreases.



Figure 6.8 : Leakage test (rel. angle 90°, A=pressure and B=measure port).

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measurement number





measurement number

Figure 6.10 : Leakage test of type 4 (rel angle 131°15', A=pressure port and B=measure port).

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Figure 6.11 : Leakage test (rel. angle 131°15', A=pressure



measurement number

Figure 6.12 : Leakage test (rel. angle 131°15', B=pressure and D=measure port).

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Torque measurement :

Figure 6.13 shows torque curves of type 1,2,3 and 4. - The torque at the beginning is higher than at the end. - Type 1,2 and 4 show a high torque at the beginning comparing to the end.



measurement number

Figure 6.13 : Torque between house and core.

Observations during blocks :

The temperature curves for all types and the room temperature are plotted in figure 6.14. - The temperature of type 4 is a few degrees higher than the rest.

All types show leakage during the blocks.
Particles of seal and backup ring (if used) appear at the outside.

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Figure 6.14 : Temperature curves.

Pressure drop :

Because type 3 is better according to drawing demands than type 3, a pressure drop test is done only to type 3.

The pressure drop Pd has to be Pd≤2.5 bars. - All measured pressure drops satisfy this demand (see table 6.1).

Rel.angle	DAF	pressret.	plugged	pressure drop(bars)
252°33'	162°33'	A-C	B-D	1.103
252°33'	162°33'	B-D	A-C	0.828
178°45'	88°45'	B-C	A-D	0.965
178°45 '	88°45'	A-D	B-C	0.828

Table 6.1 : Pressure drop to Swivel Valve type 3.

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7 CONCLUSIONS

- The basic Swivel Valve configuration type 1 satisfies regarding wear.
- The other configurations also satisfy regarding wear especially the ionic nitrided configuration.
- The geometric changes concerning pockets are acceptable (pressure drop<2.5 bars).
- All 4 configurations show leakage (the ionic nitrided configuration the most).
- The seals used don't satisfy.

8 RECOMMENDATIONS

- The supply of basic Swivel Valve configurations can continue.
- The inspection of the Swivel Valves has to be improved.
- The seal problem has to be solved.
- The ionic nitride treatment has to be investigated further.

REFERENCE LIST (person)

NAME	ACTIVITY	DEPARTMENT
Beijk, D.J.H	:chrome plating cores	:Serv.Treatm.Oper.
Cuypers, J.B.A.	:Furniture, test room	:Werkv.Niet Versp SP
Didden, C.J.	:Prod. test frame	:Ger. Beh.&Onderh.
Dielissen, C.H.	:S.V. production	:Leid.Ger.Makerij
Haans, P	:Electrical part	:Onderhoud SP
Hendriks,R.H.A.	:Incoming products	:Mat.Beh.&Ont.kontr.
Honig, J.J.	:S.V. conf. selection	:CL Chemie&Corrosie
Huberts, P.T.J.M.	:Hydraulic unit	:Manuf. Technology
Jannis, L.M.	:S.V. conf. selection	:Manuf. Technology
Kirkels, G.M.C.	:S.V. production	:Werkvoorbereiding
Kruijff, R. de	:Ion nitriding treatm.	:CL Warmtebeh.&Verm.
Martens, J.H.M.	:S.V. conf. selection	:CL Metaalkunde
Mierlo van	:Ion ntriding treatm.	:Mamesta
Nelissen, J.W.A.	:S.V. conf. selection	:Serv. Treatm. Oper.
Slippens, R.H.	:Purchasing	:Inkoop SP
Valentijn, G.N.	:Prod. test frame	:Onderhoud SP
Verboven, P.A.W.J	:Prod. test frame	:Onderhoud SP
Verhoeven, H.T.J.	:Measurement S.V.	:Fabricage Aerospace
Zalm, J. van de	:Different activities	:Werkv.Niet versp.SP
Zon, A.J.C. van	:F-16 assemblage room	:Planning Versp. SP

I SWIVEL VALVE OPERATION

The Swivel Valve takes care of a fluid reversal within the actuator. This reversal is important for the rotation of the nose landing gear, when one of the actuators passes through its dead point position (figure I.1 position 2). If the nose landing gear rotates from position 1 to 2, the actuator has to pull and when it rotates from 2 to 3 the actuator has to push. So when the nose landing gear makes a rotation movement in one direction through the dead point of the actuator, the movement of the actuator has to reverse.



Figure I.1 : Actuator motion in combination with the nose landing gear.

-I1-

The Swivel Valve takes care of the pressure supply to the actuator, reversing the direction of the actuator movement, so the nose landing gear can proceed rotating in one direction.

The house is mounted on the nose landing gear and stands still. The core is mounted on the actuator and rotates in the house. Depending on the position of the actuator relative to the gear the flow is reversed.

II DRAWINGS

Drawing List

no.

title

SP10263	:	Assembly Testing Tool (Swivel Valve)
SP10264	•	Base Plate
SP10265	:	Positioning Block (Swivel Valve)
SP10266	:	Positioning Arm
SP10267	:	Plate
SP10268	:	Bracket
SP10269	:	Bracket, Motor
SP10270	:	Shaft, Tension Pulley
SP10271	:	Consolidation Strap
SP10272	:	Plate
SP10275	:	Shaft
SP10276	:	Rotor
SP10277	:	Rotor Pin
SP10278	:	Positioning Pin
SP10279	:	Bush, Tension Pulley
SP10280	:	Block, Tension Pulley
SP10281	:	Strap, Tension Pulley
SP10283	:	Pin Tension Pulley
SP10284	:	Pin
SP10285	:	Snaarschijf
SP10287	:	Coupling, Flexible
SP10288	:	Core Experimental type 1
SP10289	:	Core Experimental type 2
SP10290	:	Core Experimental type 3
SP10291	:	Core Experimental type 4
SP10292	:	Base Plate Torque Tool
SP10293	:	Positioning Strap
SP10294	:	Torque Tool Assembly
SP10295	:	Positioning Ring for Ionic Nitriding S.V.
SP10296	:	Positioning Plate for Ionic Nitriding S.V
SP10297	:	Frame Testing Tool Swivel Valve Messier

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37 A3 S	SP 10272	1	AlMgSi1	50×12,5 × 100	PLATE	57				_						
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31	WC 08 DU	4	PTFE	Ø10/Ø20	THRUST WASHER GLACIER	51			4	0	8.8	M6 × 15	CIL.KOPSC	HR. m.b.z.		
30	FMB 1012 DU	4	PTFE	φ10/φ12 12	GLACIER METAL NED. BUSH	50	A3	SP 10	271	8	C60/04011	⊈±12 × 220	CONSOLID	ATION S	FRAP	
29 A3	SP10278	4	35NCD16 / S99	Ø18 × 417	POSITIONING PIN	49	A2	SP 10	266	4	C60/04011	45 × 12 × 380	POSITIONI	NG ARM	(45°)	
28	A 2 0	11	ST.	Ø20 AS	SEEGERRING	48				1	8.8	M6 × 20	CIL.KOPSC	HR. m.b.z.		
27	WC 18 DU	11	PTFE	Ø20/Ø36	THRUST WASHER GLACIER	47	A3	SP 10	284	1	C 60/ 0 40 11	Ø12 × 55	PIN			
26	EMB 2011 5 DU	4	PTFF	020/023 11.5	GLACIER METAL NED, BUSH	46	A	3 SP 10	283	1	IST37	Ø16 × 210	PIN, TENS	ION PULL	EY-	
25 A3	SP10277	1	C60/04011	φ 30 × 79.5	ROTOR PIN	45	+	_		2	8.8	M8 × 65	CIL.KOPSC	HR. m.b.z.		1
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21 43	SP10275	2	C60/04011	Ø35 x 148	SHAFT	41		DIN	7979	2	ST	Ø8m 6 × 60	PASPEN			
- 20	WC 22 DU	2	PTFF	Ø25/Ø40 (Ø24	THRUST WASHER GLACIER						· · · · · · · · · · · · · · · · · · ·				_	+
19	SP 10286	2	zie SP10285	DW 180 SPA	SNAARSCHIJF v.d.heijden	1										1
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17	SPA 1207	1	/	L = 1207	STEMIN (v.d heijden)V-SNAAR											
16		11	8.8	M12 ×50	CIL.KOPSCHR. m.b.z.	1										1
15 A3	SP 10280	1	C60/04011	50 × 25 × 50	BLOCK, TENSION PULLEY-											
14 43	SP10281	2	C 60 / 04011	50×10 × 60	STRAP TENSION PULLEY-	1										
13 43	SP 10 270	1	C60/04011	Ø 25h6 × 66	SHAFT TENSION PULLEY-	1										1
12	EMB 2521501	1 6	PTFE	Ø25/Ø28/21.5	GLACIER METAL NED. BUSH	1									- 	_
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III ELECTRIC MOTOR CALCULATION

The driving disc must be able to run at two velocities namely at 0.5 rot./s for test manoeuvre A and at 1.5 rot./s for test manoeuvre B.

When a transmission between the electric motor and the driving disc is used with two identical disc pairs, one driving belt can be used to switch between the discs (see figure III.1).



Figure III.1 : Transmission between electric motor and driving disc.

When using 1 driving belt and 2 identical disc pair to achieve the 2 different angular velocities for the driving disc, the relation between the diameters on the disc pairs can be calculated.

One disc pair is connected to the electric motor shaft and the other disc pair is connected to the driving disc shaft. The combination in transmission between the 2 discs is :

-III1-

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motor --- driving disc

1) small diameter --- large diameter --> 0.5 rot./s

2) large diameter --- small diameter --> 1.5 rot./s
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-III2-

with using 2 identical large diameters and 2 small diameters.

If we are using one driving belt and 2 identical disc pairs to achieve the 2 different angular velocities for the driving disc, the relation between the diameters on the disc pairs can be calculated.

One disc pair consist of 2 disc with diameter D1 and D2. nD1 and nD2 are the angular velocities of the driving disc with values : nD1=1.5 rotations/s

nD2=0.5 rotations/s.

Because of the same velocities at the edge of the disc 1 and disc 2 of the disc pair and nm is the motor angular velocity then :

 $v1 = v2 = nm x D1 = nD1 x D2 \rightarrow nD1 = nm x D1/D2 = 1.5$ and $v2 = v1 = nm x D2 = nD2 x D1 \rightarrow nD2 = nm x D2/D1 = 0.5$

> so D1/D2=/3 and nm=0.87 rotations/s

So the needed angular velocity for the electric motor is 0.87 rotations per second.

The following diameters for the discs have been selected : D1=180mm D2=112mm

From earlier tests it is known the maximum torque between house and core of the Swivel Valve is 36Nm. Building in a

safety factor of 2 the torque is 72Nm. Considering this torque and considering testing for 4 configurations at once, the power needed can be calculated. It is assumed that the absolute torque in the house is constant maximum (see figure III.2).



figure III.2 : Power used for 4 Swivel Valve configurations versus rotation driving disc.

Figure III.2 shows that the maximum power is 477.98 Watts.

Regarding this it was decided to choose an Electrim Electric motor with build on reduction case : electric motor : Kollmer SKG 90S4 1.1 KW 1415r 2/3V 50Hz B5, Ip-54, iso-F reduction gear : RF2/63 1:25 case N2=56rot./min. B5 flens construction, axle 35x70mm servo factor 2:1

-III3-

So the angular velocities for the driving disc will be : nD1 = nm x D2/D1 = 1.5 rotations/s nD2 = nm x D1/D2 = 56/60 x 112/180 = 0.58 rotations/s