

Traveling waves in a finite condensation rate model for steam injection

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TRAVELING WAVES IN A FINITE CONDENSATION RATE MODEL FOR STEAM INJECTION

J. BRUINING AND C.J. VAN DUIJN

ABSTRACT. Steam drive recovery of oil is an economical way of producing oil even in times of low oil prices and is used world wide. This paper focuses on the one-dimensional setting, where steam is injected into a core initially containing oil and connate water while oil and water are produced at the other end. A three phase (oil, water, steam) hot zone develops, which is abruptly separated from the two phase (oil+water) cold zone by the steam condensation front. The oil, water and energy balance equations (Rankine-Hugoniot conditions) cannot uniquely solve the system of equations at the steam condensation front. In a previous study we showed that two additional constraints follow from an analysis of the traveling wave equation representing the shock, however within the shock we assumed local thermodynamic equilibrium. Here we extend the previous study and include finite condensation rates, using that appropriate scaling requires that the Peclet number and the Damkohler number are of the same order of magnitude. We give a numerical proof, using a color coding technique, that given the capillary diffusion behavior and the rate equation a unique solution can be obtained. It is proven analytically that the solution for large condensation rates tends to the solution obtained assuming local thermodynamic equilibrium. Computations with realistic values to describe the viscous and capillary effects show that the condensation rate can have a significant effect on the global saturation profile e.g. the oil saturation just upstream of the steam condensation front.

1. INTRODUCTION

Steam injection, as a method for enhanced oil recovery, received considerable attention in the petroleum engineering literature during past decades ([11], [16], [32]). Recently there is renewed interest for the purpose of removing oil spills from the subsurface ([3], [19], [20], [23], [37], [38]). The processes involved are extremely complex and pose challenging questions concerning theory ([12], [26], [45], [46]) experiment ([13], [24], [43]) and numerical modeling ([1], [5], [8], [11], [29], [30], [31]).

The theoretical study described in this paper builds on previous work ([7], [39]). In [7] we considered a simplified model describing oil recovery by steam drive. The proposed model assumes small capillary forces and instantaneous condensation as a result of thermodynamic equilibrium. It has an upstream hot three-phase (oil, water, steam) flow zone and a downstream cold two-phase (oil, water) flow zone. The upstream and downstream zones are separated by a relatively thin transition region, which is described by a (local) steam condensation/capillary diffusion model based on the ideas of Udell et al. ([28], [40], [42]). In the limit of zero capillary forces the transition region collapses to form a steam condensation front (SCF). Disregarding capillary pressure away from the condensation front, a 2x2 hyperbolic system ([14], [15]) arises for water and steam. This system cannot be solved uniquely without additional conditions at the SCF. To find these conditions we studied traveling waves

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of the capillary model in the transition region ([21], [22], [25], [27]). In [7], we investigated the effect of different capillary pressure behavior, the effect of gradual versus abrupt temperature decline from the steam temperature to ambient temperature and the effect of non-zero gas saturation at the SCF. In all these cases we assumed local thermodynamic equilibrium. We found and made explicit that details of the transition model affect the global behavior of the steam displacement process. It is therefore of interest to investigate whether the global behavior also depends on the rate constant i.e. if we drop the assumption of thermodynamic equilibrium and implement a condensation rate model. A finite reaction rate model is also preferred in numerical simulations. We expect that in the limit of large rate constants the results for local equilibrium are retrieved. The aim of this paper is to investigate those two aspects.

In Section 2 we briefly describe the model and recall the model equations. The hyperbolic setting and a summary of previous results is given in Section 3. In Section 4 we define the traveling wave problem and the method to obtain the unique solution for a given condensation model. The route to thermodynamic equilibrium is explained in Section 5 by sending the rate constant to infinity. We end in Section 6 with computations for some realistic cases and comparison to previous results.

2. FINITE RATE CONDENSATION MODEL

2.1. Physical considerations. Oil displacement by steam drive through a porous medium is a complex physical process which is controlled by the steam condensation process and by viscous and capillary forces, see for instance Stewart & Udell [40] and Wingard & Orr [44]. Following ideas of Shutler [39], we proposed in [7] a one-dimensional model where all complexity is confined to a small transition region in which the condensation occurs and capillary forces act. The model describes the case of injecting steam in a linear core originally filled with oil and connate water. The porosity φ and permeability k are constant. We allow for temperature dependent liquid viscosities except that we assume the steam viscosity to be independent of temperature, because these viscosities are small anyway and the temperature dependence of $\mu_g \sim T^{0.6}$ is much smaller than for the liquid viscosities.

The core is horizontal and we disregard the effects of gravity. Transverse capillary pressure diffusion is sufficiently large to guarantee a uniform saturation over the cross-section. The core is positioned along the positive x-axis with flow from left to right, implying that all variables are functions of position x , and time t . The displacement is considered to occur at constant pressure, in the sense that we disregard flow induced pressure gradient effects on the thermodynamic properties, reaction rates, fluid densities and viscosities. Therefore the pressure does not explicitly enter in the model equations but it determines the value of some parameters. The temperature dependence of the parameters is summarized in the table. The oil considered is dead oil: i.e. it does not occur in the gas phase. Dissolution of liquid oil in water and vice versa is disregarded. The condensation occurs between an upstream three-phase flow zone at steam temperature T_b where oil, water and steam are present and a downstream two-phase flow zone at the initial reservoir temperature T_o where water and oil are present. In the upstream and downstream regions capillary forces are disregarded. Consequently these regions are adequately described by an (extended) Buckley-Leverett approach. We use powerlaw relative permeabilities (both quadratic and fourth powers), as well as Stone I expressions [9].

Table: Summary of physical input parameters ¹			
<i>Physical quantity</i>	<i>symbol</i>	<i>value</i>	<i>unit</i>
characteristic length	L	100	[m]
steam temperature	T_b	486	[K]
reservoir temperature	T_o	313	[K]
injection rate steam	u_{inj}	$9.52 \cdot 10^{-4}$	$[m^3/m^2/s]$
steam viscosity	μ_g	$1.63 \cdot 10^{-5}$	[Pa s]
oil viscosity at T_b	$\mu_o(T_b)$	$2.45 \cdot 10^{-3}$	[Pa s]
oil viscosity at T_o	$\mu_o(T_o)$	0.180	[Pa s]
water viscosity at T_b	$\mu_w(T_b)$	$1.30 \cdot 10^{-4}$	[Pa s]
water viscosity at T_o	$\mu_w(T_o)$	$7.21 \cdot 10^{-4}$	[Pa s]
reference viscosity	μ_w^*	$5.0 \cdot 10^{-4}$	[Pa s]
Brooks-Corey sorting factor	λ_s	2	[-]
Rate constant	q_b	10^3	$[s^{-1}]$
enthalpy $H_2O(l)(T_o) \rightarrow H_2O(g)(T_1)$	ΔH	2636	[kJ/kg]
effective heat capacity of rock	$(\rho c)_r$	2029	$[kJ/m^3/K]$
thermal coefficient	α	0.017	[-]
capillary diffusion constant	D	$1.85 \cdot 10^{-7}$	$[m^2/s]$
diffusion correction factor	d	10^{-2}	[-]
velocity SCF	v_{st}	$7.12 \cdot 10^{-5}$	$[m/s]$
porosity	φ	0.38	$[m^3/m^3]$
permeability	k	$1.0 \cdot 10^{-12}$	$[m^2]$
interfacial tension	σ	0.03	[N/m]
water density	ρ_w	1000	$[kg/m^3]$
steam density	ρ_g	10.2	$[kg/m^3]$
connate water saturation	S_{wc}	0.15	$[m^3/m^3]$
residual gas saturation	S_{gr}	0.0	$[m^3/m^3]$
residual oil saturation	S_{or}	0.0	$[m^3/m^3]$

All condensation occurs in a thin region called the steam condensation front (SCF). The constant travel speed of the SCF is determined from an energy balance that is decoupled from the mass balance equations. The decoupling is achieved by disregarding the effect of fluid content on the heat capacity $(\rho c)_r$ of the porous medium. The velocity v_{st} is determined from a local heat balance, in which the heat released by the condensing steam impinging on the SCF is equal to the amount of heat necessary to warm up the reservoir, see Mandl and Volek [26]. The result is

$$v_{st} = \frac{\rho_g \Delta H u_{inj}}{(\rho c)_r (T_b - T_o)} .$$

The symbols appearing in this expression are explained in the table.

Within the transition region there is an interplay between viscous forces, capillary forces, and the condensation process. In this paper we use a finite rate condensation model. *There are three dimensionless numbers involved in the processes that occur in the transition zone*

¹The values of the steam parameters in the table assume a steam pressure of 20 bar. Furthermore the value of the thermal coefficient α is based on a thermal diffusivity of $9.85 \cdot 10^{-7} [m^2/s]$. Note that this coefficient is proportional to the ratio of the capillary and thermal diffusivity.

i.e. the Peclet number for mass transport Pe , the Peclet number for heat transport Pe_T and the Damkohler number Da . These numbers indicate respectively the ratio of phase transport by convection and diffusion, the ratio of heat convection and thermal conductivity and the ratio of the rate of convected phase transport and the condensation rate. In the model we assume an instantaneous temperature drop from steam temperature to reservoir temperature as the steam saturation becomes zero. Furthermore we assume that the Damkohler number Da and the Peclet number Pe are of the same order of magnitude. We compare our results to those obtained in [7], where it was assumed that all steam condenses at a single point in the transition region, the actual (SCF). The rate of condensation is sufficiently fast so that indeed all condensation occurs in a small neighborhood of the steam condensation front. Here "small" must be understood in a suitable dimensionless context. In the condensation zone the temperature drops from steam temperature T_b to the original reservoir temperature T_o and steam condenses at a rate proportional to $(T_b - T)$, where T is the prevailing temperature. As long as there is steam, the condensation rate is proportional to the saturation S_g . When the steam saturation is zero, the pores are fully saturated with water and oil, and the condensation rate becomes zero. This leads to the following expression for the mass condensation rate q

$$q = \begin{cases} \rho_g q_b \frac{T_b - T}{T_b - T_o} S_g & \text{for } T \leq T_b, \quad 0 < S_g \leq 1 - S_{wc}, \\ 0 & \text{otherwise} \end{cases}, \quad (2.1)$$

where q_b is the condensation rate parameter. As in [7] we assume that the temperature distribution can be determined independently from the condensation process. In fact, in [7] we distinguished between an exponential decline and a step wise decline. In this paper we confine ourselves to the step-wise decline. Hence the temperature is discontinuous and given by

$$T = \begin{cases} T_b & x < v_{st}t, \\ T_o & x \geq v_{st}t. \end{cases}$$

The phase densities ρ_α ($\alpha = w, o, g$) are assumed to be constant throughout this paper.

2.2. Model equations. The mass balance equations for water, steam and oil read

$$\varphi \frac{\partial(\rho_w S_w)}{\partial t} + \frac{\partial(\rho_w u_w)}{\partial x} = q, \quad (2.2)$$

$$\varphi \frac{\partial(\rho_g S_g)}{\partial t} + \frac{\partial(\rho_g u_g)}{\partial x} = -q, \quad (2.3)$$

$$\varphi \frac{\partial(\rho_o S_o)}{\partial t} + \frac{\partial(\rho_o u_o)}{\partial x} = 0. \quad (2.4)$$

where q is given by expression (2.1). The phase saturations satisfy

$$0 \leq S_{wc} \leq S_w \leq 1, \quad 0 \leq S_g, S_o \leq 1 - S_{wc}, \quad (2.5)$$

in other words we assume there is no residual oil and gas in the system. We use Darcy's law for multi-phase flow to express all phase velocities u_α in terms of the total velocity u and the capillary pressures, see [1] and also ([10], [18], [33], [34]). This gives

$$u_w = u f_w + \lambda_o f_w \frac{\partial p_{c,ow}}{\partial x} + \lambda_g f_w \frac{\partial p_{c,gw}}{\partial x}, \quad (2.6)$$

$$u_g = u f_g - \lambda_o f_g \frac{\partial p_{c,go}}{\partial x} - \lambda_w f_g \frac{\partial p_{c,gw}}{\partial x}, \quad (2.7)$$

$$u_o = u f_o - \lambda_w f_o \frac{\partial p_{c,ow}}{\partial x} + \lambda_g f_o \frac{\partial p_{c,go}}{\partial x}. \quad (2.8)$$

Here

$$u = u_w + u_o + u_g, \quad (2.9)$$

$p_{c,\alpha\beta}$ the capillary pressure, being the pressure difference between phase α and phase β , and f_α the fractional flow function

$$f_\alpha = \frac{\lambda_\alpha}{\lambda_o + \lambda_w + \lambda_g}. \quad (2.10)$$

Further λ_α denotes the mobility of phase α , given by

$$\lambda_\alpha = \frac{k k_{r\alpha}}{\mu_\alpha},$$

where $k_{r\alpha}$ is the relative permeability (fourth powers of the effective saturations) and μ_α the phase viscosity. Since water and oil experience different temperatures, their viscosities may vary significantly across the steam condensation front. Realistic values ([4], [35], [41]) are given in the table. In later sections we use the notation f_α^\pm , where f_α^- denotes the fractional flow function in the hot steam zone and f_α^+ in the cold oil zone.

Since

$$\sum_\alpha S_\alpha = \sum_\alpha f_\alpha = 1 \text{ and } u = \sum_\alpha u_\alpha,$$

we can eliminate, for instance, S_o from the equations. Further summing equations (2.6), (2.7) and (2.8) and using equation (2.9), we find

$$\frac{\partial u}{\partial x} = -\frac{1}{\rho_g} \left(1 - \frac{\rho_g}{\rho_w}\right) q = -q_b \left(1 - \frac{\rho_g}{\rho_w}\right) \left(\frac{T_b - T}{T_b - T_o}\right) S_g. \quad (2.11)$$

Thus our primary variables are u , S_w and S_g for which we have equations (2.11), (2.2+2.6) and (2.3+2.7). Injecting only steam from the left at $x = 0$ and having only oil and connate water present at $t = 0$, requires the boundary/initial conditions:

$$u(0, t) = u_{inj}, \quad S_w(0, t) = S_{wc}, \quad \text{and } S_g(0, t) = 1 - S_{wc}, \quad (2.12)$$

for all $t > 0$ and

$$S_w(x, 0) = S_{wc} \text{ and } S_g(x, 0) = 0, \quad (2.13)$$

for all $x > 0$. In (2.12), u_{inj} denotes the injection velocity of the steam.

We want to write the Darcy velocities u_w and u_g in terms of capillary diffusion involving S_w and S_g only. For this purpose we note that

$$p_{c,gw} = p_g - p_w = p_{c,go} + p_{c,ow},$$

where $p_{c,ow} = p_{c,ow}(S_w)$ is a strictly decreasing function of the water saturation and where $p_{c,go} = p_{c,ow}(1 - S_g)$ is a strictly increasing function of the gas saturation. For instance in [7] we considered the Brooks-Corey expression, see also ([2], [6], [36]),

$$p_{c,ow} = \frac{\sigma}{2} \sqrt{\frac{\varphi}{k}} \left(\frac{\frac{1}{2} - S_{wc}}{1 - S_{wc}} \right)^{1/\lambda_s} \left(\frac{S_w - S_{wc}}{1 - S_{wc}} \right)^{-1/\lambda_s}, \quad (2.14)$$

where $\lambda_s > 0$ is the sorting factor.

Using these observations we obtain

$$u_w = u f_w - \mathcal{D}_{ww} \frac{\partial S_w}{\partial x} - \mathcal{D}_{wg} \frac{\partial S_g}{\partial x}, \quad (2.15)$$

$$u_g = u f_g - \mathcal{D}_{gw} \frac{\partial S_w}{\partial x} - \mathcal{D}_{gg} \frac{\partial S_g}{\partial x}, \quad (2.16)$$

where

$$\begin{aligned} \mathcal{D}_{ww} &= -(\lambda_o + \lambda_g) f_w \frac{dp_{c,ow}}{dS_w} > 0, \\ \mathcal{D}_{wg} &= -\lambda_g f_w \frac{dp_{c,go}}{dS_g} < 0, \\ \mathcal{D}_{gw} &= \lambda_w f_g \frac{dp_{c,ow}}{dS_w} < 0, \\ \mathcal{D}_{gg} &= (\lambda_o + \lambda_w) f_g \frac{dp_{c,go}}{dS_g} > 0. \end{aligned} \quad (2.17)$$

Except in Section 6, where we work out a realistic case based on expression (2.14), we consider throughout this paper

$$\begin{aligned} \mathcal{D}_{ww} &= \mathcal{D}_{gg} = \mathcal{D} = \text{constant}, \\ \mathcal{D}_{wg} &= \mathcal{D}_{gw} = 0, \end{aligned}$$

where the constant diffusivity is given by

$$\mathcal{D} = \frac{\sigma \sqrt{\varphi k}}{\mu_w^*} d.$$

Here μ_w^* denotes a characteristic water viscosity *e.g.* $\mu_w(T_o)$ and d accounts for the effect of the water relative permeability and the functional relation of the capillary pressure.

2.3. Rescaled equations. We rewrite the equations in dimensionless form by setting

$$S_w := \frac{S_w - S_{wc}}{1 - S_{wc}}, \quad S_o := \frac{S_o}{1 - S_{wc}}, \quad S_g := \frac{S_g}{1 - S_{wc}}, \quad (2.18)$$

$$T := \frac{T - T_o}{T_b - T_o}, \quad u := \frac{u}{u_{inj}}, \quad x := \frac{x}{L}, \quad t := \frac{u_{inj} t}{\varphi L}, \quad (2.19)$$

where L represents a characteristic length of the problem, for instance the distance between injection and production point. Introducing the reciprocal Peclet number $\varepsilon := \mathcal{D}/(u_{inj} L)$ and the dimensionless rate constant $r := q_b \mathcal{D}/u_{inj}^2$ we find

$$\frac{\partial u}{\partial x} = -\frac{r}{\varepsilon}(1-\alpha)(1-T)S_g, \quad (2.20)$$

$$\frac{\partial S_w}{\partial t} + \frac{\partial u f_w}{\partial x} = \alpha \frac{r}{\varepsilon}(1-T)S_g + \varepsilon \frac{\partial^2 S_w}{\partial x^2}, \quad (2.21)$$

$$\frac{\partial S_g}{\partial t} + \frac{\partial u f_g}{\partial x} = -\frac{r}{\varepsilon}(1-T)S_g + \varepsilon \frac{\partial^2 S_g}{\partial x^2}. \quad (2.22)$$

Here $\alpha = \rho_g/\rho_w$. Generally $\alpha \ll 1$. The boundary and initial conditions follow from (2.12) and (2.13). They read

$$u(0, t) = 1, \quad S_w(0, t) = 0, \quad S_g(0, t) = 1 \quad \text{for all } t > 0 \quad (2.23)$$

and

$$S_w(x, 0) = S_g(x, 0) = 0 \quad \text{for all } x > 0. \quad (2.24)$$

The scaled temperature $T = T(x, t)$ is discontinuous at the SCF with

$$T = \begin{cases} 1 & x < vt, \\ 0 & x \geq vt, \end{cases} \quad (2.25)$$

where $v = v_{st}/u_{inj}$.

Remark 2.1. Direct scaling of equation (2.11) gives

$$\frac{\partial u}{\partial x} = -\frac{q_b L}{u_{inj}}(1-\alpha)(1-T)S_g$$

with $\frac{q_b L}{u_{inj}}$ being the dimensionless rate parameter (Damkohler number). To balance terms in the equations, in particular after the additional blow up $x := x/\varepsilon$, we need

$$\frac{q_b L}{u_{inj}} = \frac{r}{\varepsilon}, \quad \text{with } r = O(1) \text{ as } \varepsilon \downarrow 0.$$

Realistic numbers from the table give:

$$\frac{q_b L}{u_{inj}} \sim 10^8, \quad r \sim 10^2 \text{ and } \varepsilon \sim 10^{-6}.$$

3. HYPERBOLIC SETTING AND PREVIOUS RESULTS

When the scale L of the problem is such that $\mathcal{D} \ll u_{inj}L$, and thus $\varepsilon \ll 1$, one expects that equations (2.20)-(2.22) reduce to a hyperbolic system, similar to the one studied in [7], in which all steam condenses at the steam condensation front: i.e.

$$\frac{\partial u}{\partial x} = -(1-\alpha)\Lambda\delta(x-vt), \quad (3.1)$$

$$\frac{\partial S_w}{\partial t} + \frac{\partial u f_w}{\partial x} = \alpha\Lambda\delta(x-vt) \quad , \quad (3.2)$$

$$\frac{\partial S_g}{\partial t} + \frac{\partial u f_g}{\partial x} = -\Lambda\delta(x-vt) \quad . \quad (3.3)$$

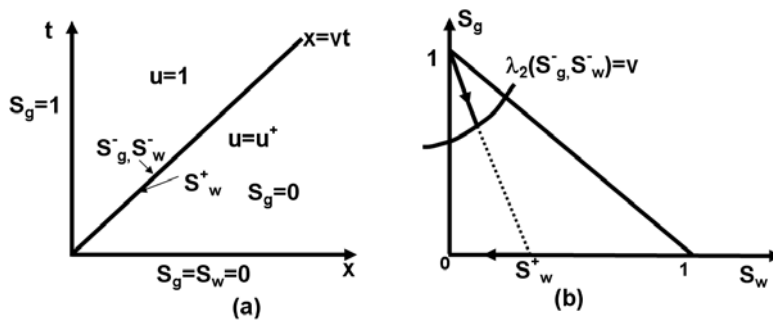


FIGURE 3.1. Water and gas saturations in hyperbolic model ((3.1)-(3.3)). In (a): saturations in the (x, t) plane with unknowns $(S_w^-, S_g^-, S_w^+, u^+)$ at the steam condensation front. In (b) the saturation path in the (S_w, S_g) plane.

Here Λ is the condensation rate that originates from the condensation terms in equations (2.20)-(2.22) and δ the Dirac measure at $x = vt$. Below we make this precise for traveling wave solutions.

In [7] we showed that a solution of the system (3.1)-(3.3), satisfying (2.23) and (2.24) can only be a fast rarefaction with $u = 1$ in the steam region $\{x < vt\}$, with a shock at the steam condensation front $\{x = vt\}$, and with two-phase Buckley-Leverett behavior in the cold region $\{x > vt\}$ where $S_g = 0$, see Figure 3.1

We also showed that in order to obtain a uniquely constructed solution, a local transition model at the steam condensation front must be introduced. In the hyperbolic setting there are 4 unknowns at the steam condensation front: the water and gas saturation (S_w^-, S_g^-) on the left, and the water saturation S_w^+ and Darcy velocity u^+ on the right. Counting the number of equations between them we have from mass conservation two Rankine-Hugoniot conditions

$$(RH) \begin{cases} f_w^- - vS_w^- + \alpha\Lambda = u^+f_w^+ - vS_w^+, \\ f_g^- - vS_g^- - \Lambda = 0, \end{cases} \quad (3.4)$$

with $\Lambda = (1 - u^+) / (1 - \alpha)$. A third equation follows from the condition that the fast rarefaction has to match up with the velocity of the steam condensation front. This gives the entropy condition

$$(E) \quad \lambda_2(S_w^-, S_g^-) = v, \quad (3.5)$$

where λ_2 denotes the largest eigenvalue at (S_w^-, S_g^-) of the Jacobian matrix of the saturation flux (f_w, f_g) . The missing-fourth-relation follows from a traveling wave analysis of the transition model. In fact, the existence condition for a traveling wave, giving the shock its viscous profile, enabled us to construct a unique shock solution. We considered several local transition models, all having instantaneous condensation, and investigated their influence on the global solution through the shock condition at the steam condensation front.

The main purpose of this paper is to understand the finite rate condensation model proposed in Section 2. We do this by analyzing traveling wave solutions of equations (2.20)-(2.22). Such solutions allow us to quantify the role of the rate parameter r and can be used as a building block in the construction of a unique shock solution.

In the finite rate condensation model the rate constant is r/ε , where $r = O(1)$ and ε is small. This is chosen to balance terms in the equations. To see this we set

$$\eta = \frac{x - vt}{\varepsilon} \quad (3.6)$$

and consider the waves

$$S_i = S_i(\eta) \quad (i = w, g) \quad \text{and} \quad u = u(\eta). \quad (3.7)$$

Because of (2.25) we have $T = 1 - H(\eta)$, where H denotes the Heaviside function

$$H(\eta) = \begin{cases} 0 & \eta < 0 \\ 1 & \eta > 0 \end{cases}$$

Substitution of (3.6) and (3.7) into (2.20)-(2.22) results in the system of ordinary differential equations

$$\left. \begin{aligned} u' &= -(1 - \alpha) r (1 - T) S_g, \\ -v S_w' + (u f_w)' &= \alpha r (1 - T) S_g + S_w'', \\ -v S_g' + (u f_g)' &= -r (1 - T) S_g + S_g'', \end{aligned} \right\} \quad (3.8)$$

where the primes denote differentiation with respect to η . Note that ε has disappeared from the formulation and that the domain of the equations, for $\varepsilon \downarrow 0$, ranges from $\eta \rightarrow -\infty$ to $\eta \rightarrow +\infty$. As in [7] we impose the boundary conditions

$$(BC) \begin{cases} S_w(-\infty) = S_w^-, & S_g(-\infty) = S_g^-, & u(-\infty) = 1 \\ S_w(\infty) = S_w^+, & S_g(\infty) = S_g^+, & u(\infty) = u^+, \end{cases}$$

that satisfy (RH), with Λ appropriately chosen, and (E).

Suppose, for the moment, that a traveling wave exists and that the decay of $S_g(\eta) \rightarrow 0$ as $\eta \rightarrow \infty$ is such that

$$\int_0^{\infty} S_g(\eta) d\eta < \infty \quad (3.9)$$

and

$$\lim_{\varepsilon \downarrow 0} \frac{1}{\varepsilon} S_g\left(\frac{\delta}{\varepsilon}\right) = 0 \quad \text{for every } \delta > 0. \quad (3.10)$$

Clearly, for $i = w, g$,

$$S_i^\varepsilon(x, t) = S_i\left(\frac{x - vt}{\varepsilon}\right) \quad \text{and} \quad u^\varepsilon(x, t) = u\left(\frac{x - vt}{\varepsilon}\right)$$

are solutions of equations (2.20)-(2.22). In these equations the condensation terms satisfy for any $t > 0$

$$\frac{r}{\varepsilon} \int_{\mathbb{R}} (1 - T(x, t)) S_g^\varepsilon(x, t) dx = \frac{r}{\varepsilon} \int_{vt}^{\infty} S_g^\varepsilon(x, t) dx = r \int_0^{\infty} S_g(\eta) d\eta$$

for all $\varepsilon > 0$, and

$$\lim_{\varepsilon \downarrow 0} \frac{r}{\varepsilon} (1 - T(x, t)) S_g^\varepsilon(x, t) = 0 \quad \text{for all } x \neq vt.$$

Hence

$$\lim_{\varepsilon \downarrow 0} \frac{r}{\varepsilon} (1 - T(x, t)) S_g^\varepsilon(x, t) = \Lambda \delta(x - vt),$$

where

$$\Lambda = \Lambda(r) := r \int_0^{\infty} S_g(\eta) d\eta. \quad (3.11)$$

Remark 3.1. Conditions (3.9) and (3.10) follow directly from the construction of a solution. In Section 4 we show that in the (u, S_w, S_g) -space the boundary point $(u^+, S_w^+, 0)$ is a saddle with two positive and one negative eigenvalue. The latter provides exponential decay from which (3.9) and (3.10) directly follow.

In the saturation equations from (3.8) we substitute

$$r(1-T)S_g = -\frac{1}{(1-\alpha)}u'.$$

Integrating the resulting expressions gives

$$(TW) \begin{cases} u' = -(1-\alpha)r(1-T)S_g, \\ S_w' = uf_w - vS_w + \frac{\alpha}{1-\alpha}u - (f_w^- - vS_w^- + \frac{\alpha}{1-\alpha}), \\ S_g' = uf_g - vS_g - \frac{1}{1-\alpha}u - (f_g^- - vS_g^- - \frac{1}{1-\alpha}), \end{cases}$$

with $-\infty < \eta < \infty$.

We analyze this dynamical system in the next section. To emphasize the construction and to avoid technical details, we consider the rather academic case in which the viscosities do not depend on temperature and in which the viscosity ratio's are unity. Taking in addition quadratic relative permeabilities, the fractional flow functions simplify to

$$f_i = f_i(S_w, S_g) = \frac{S_i^2}{S_w^2 + S_g^2 + (1 - S_w - S_g)^2} \quad (3.12)$$

and

$$f_i^{\pm} = f_i(S_w^{\pm}, S_g^{\pm}) \quad . \quad (3.13)$$

The results for a realistic case, with temperature dependent viscosities, a large viscosity contrast, and different relative permeabilities, will be presented and discussed in Section 6.

4. CONSTRUCTION OF SHOCKS BY MEANS OF TRAVELING WAVES

In this section we explain the construction of a unique solution satisfying the simplified dynamic system (TW) , with fractional flow functions given by (3.12) and (3.13), subject to boundary conditions (BC) satisfying the constraints of Rankine Hugoniot (RH) and entropy (E) . The construction uses a shooting argument in the three-dimensional (u, S_w, S_g) -space.

Since

$$S_w + S_g = 1 - S_o \leq 1 \quad \text{and} \quad u' \leq 0,$$

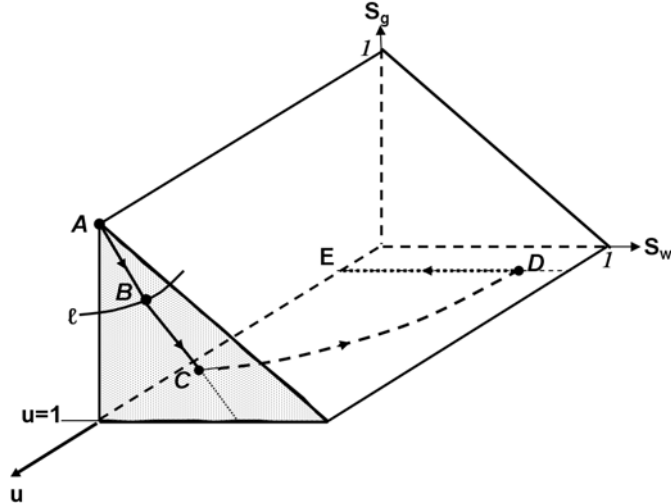
we expect that a solution is confined to the domain

$$\mathcal{R} = [u^+, 1] \times \mathbf{T}$$

where \mathbf{T} denotes the saturation triangle

$$\mathbf{T} = \{(S_w, S_g) : S_w, S_g \geq 0 \text{ and } S_w + S_g \leq 1\}$$

As in [7], the solution of the hyperbolic system (3.1)-(3.3), satisfying boundary and initial conditions (2.23 and 2.24), respectively, follows a path as AE in \mathcal{R} . This is sketched in Figure 4.1. Here A denotes the boundary condition $(1, 0, 1)$. The part AB reflects the fast


 FIGURE 4.1. Orbit in (u, S_w, S_g) space

rarefaction in the steam region where $T = 1$ and thus $u = 1$. The point $B = (1, S_w^-, S_g^-) \in l$, where

$$l = \{(u, S_w, S_g) : u = 1 \text{ and } \lambda_2(S_w, S_g) = v\}$$

Thus for points in l , the speed of the fast rarefaction and the steam condensation front coincide. With respect to the two-dimensional triangle $\{u = 1\} \times \mathbf{T}$, the point B is a non-hyperbolic saddle with eigenvalues (at (S_w^-, S_g^-) of the Jacobian matrix of the vector $(f_w - vS_w, f_g - vS_g)$)

$$e_1 < e_2 = \lambda_2 - v = 0.$$

The part BCD reflects the traveling wave as the viscous profile of the shock from B to $D = (u^+, S_w^+, 0)$. Since $T(\eta) = 1$ for $\eta < 0$, the part of the traveling wave with $-\infty < \eta < 0$ has $u = 1$ and is therefore confined to the face triangle $\{u = 1\} \times \mathbf{T}$. At C the temperature drops from boiling point to reservoir temperature implying that $T(\eta) = 0$ for $\eta > 0$. The path or orbit representing the solution now moves into the domain \mathcal{R} with strictly decreasing u . At D all steam has condensed. A two-phase Buckley-Leverett finally connects D to the initial condition $E = (u^+, 0, 0)$, with only movable oil being present.

The aim is now to show that for given $\alpha \in (0, 1)$ and $v, r > 0$, being the only parameters in the simplified problem, there exists a unique solution of (TW) which flows from $B \in l$ as $\eta \rightarrow -\infty$, through C at $\eta = 0$, to D as $\eta \rightarrow \infty$. In this solution, B and D are related by conditions (RH) .

We first consider the construction for $\eta < 0$. Since $u(\eta) = 1$ for all $\eta < 0$, we drop u from the notation. With reference to Figure 4.2, let S_w^{\max} denote the maximum water saturation for which $(S_w, S_g) \in l$ in the saturation triangle \mathbf{T} . For $N \in \mathbb{N}$ sufficiently large, let

$$S_w^-(n) = \frac{n}{N} S_w^{\max} \quad n = 0, 1, 2, \dots, N, \quad (4.1)$$

denote a uniform partition of the interval $[0, S_w^{\max}]$ and let $B(n) := (S_w^-(n), S_g^-(n))$ denote the corresponding partition of l . For each $n \in \{0, 1, \dots, N\}$ we determine at $B(n)$ the eigenvector \vec{e}_2 corresponding to the eigenvalue $e_2 = 0$. This vector is indicated in Figure

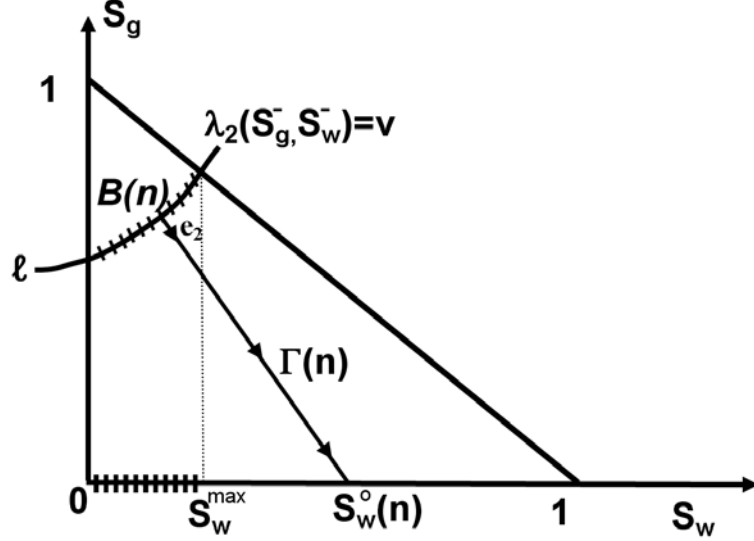


FIGURE 4.2. Construction of solution for $\eta < 0$, with $u = 1$. Here $B(n)$ denotes the point $(S_w^-(n), S_g^-(n))$

4.2. In its direction we solve the two saturation equations from (TW) with $u = 1$, kept fixed. The corresponding solution is represented by the orbit $\Gamma(n)$ in Figure 4.2. It reaches $(S_w = S_w^0(n), S_g = 0)$ at finite η . Later on we shall redefine η such that $\eta = 0$ corresponds to point C in Figure 4.1.

Next we construct the solution for $\eta > 0$. For given $n \in \{0, \dots, N\}$ we first determine from (RH) the corresponding water saturation $S_w^+(n)$ and the downstream fluid discharge $u^+(n)$. This yields the point $D(n) = \{u^+(n), S_w^+(n), 0\}$ as indicated in Figure 4.3.

Again we use (RH) , now to put equations (TW) , for $\eta > 0$, in the form

$$(TW_+) \begin{cases} u' = -(1 - \alpha) r S_g, \\ S_w' = u f_w - v S_w + \frac{\alpha}{1 - \alpha} u - F_w^+(n), \\ S_g' = u f_g - v S_g - \frac{1}{1 - \alpha} u - F_g^+(n), \end{cases}$$

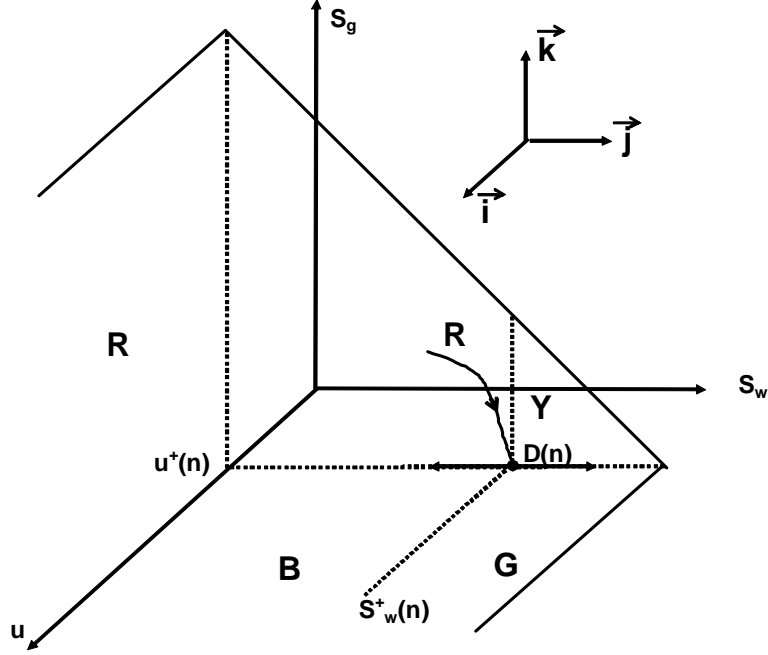
where

$$\begin{aligned} F_w^+(n) &= u^+(n) f_w(S_w^+(n), 0) - v S_w^+(n) + \frac{\alpha}{1 - \alpha} u^+(n), \\ F_g^+(n) &= -\frac{1}{1 - \alpha} u^+(n). \end{aligned}$$

We first determine the nature of the stationary point $D(n)$. From the Jacobian matrix at $D(n)$ we find one negative and two positive eigenvalues:

$$\begin{aligned} \lambda_1 &= -\frac{v}{2} - \frac{1}{2} \sqrt{v^2 + 4r} < 0 \\ \lambda_2 &= -\frac{v}{2} + \frac{1}{2} \sqrt{v^2 + 4r} > 0 \\ \lambda_3 &= u^+(n) \frac{\partial f_w}{\partial S_w}(S_w^+(n), 0) - v > 0 \end{aligned}$$

The latter being positive follows directly from a consistency condition: all characteristic speeds in the two-phase Buckley-Leverett regime must exceed the speed of the steam condensation front, see also [7]. Only the negative eigenvalue λ_1 is relevant and we have to


 FIGURE 4.3. Sketch of exit sets near $D(n) = (u^+(n), S_w^+(n), 0)$

verify that the corresponding eigenvector \vec{e}_1 points into domain \mathcal{R} . Indeed a straightforward computation gives

$$\frac{\vec{e}_1 \cdot \vec{k}}{\vec{e}_1 \cdot \vec{i}} = \frac{1}{2} \frac{v + \sqrt{v^2 + 4r}}{(1 - \alpha)r} > 0 \quad \text{for all } n \in \{0, 1, \dots, N\}.$$

Here \vec{i} and \vec{k} are unit vectors as indicated in Figure 4.3. This inequality shows that \vec{e}_1 points in the direction of increasing u and S_g . Part of the orbit representing the solution (TW_+) is sketched in Figure 4.3.

Remark 4.1. An eigenvector corresponding to λ_3 is $\vec{e}_3 = (0, 1, 0)$. Indeed, a solution of (TW_+) is $(u = u^+(n), S_w, S_g = 0)$ with S_w satisfying

$$S_w' = u^+ (f_w(S_w) - f_w(S_w^+)) - v(S_w - S_w^+).$$

Let us now turn to the full solution in \mathcal{R} . For a fixed n , with corresponding curve $\Gamma(n)$ we solve equations (TW_{+-}) with points from $\Gamma(n)$ as initial condition. Since $u' < 0$, the solution orbit will move into \mathcal{R} . As a first observation we note that any such orbit cannot leave \mathcal{R} through the side

$$\mathbf{S} = \{(u, S_w, S_g) : u^+ < u < 1, S_w + S_g = 1\}.$$

Proposition Any solution $(u(\eta), S_w(\eta), S_g(\eta))$ of (TW_+) that belongs to the interior of \mathcal{R} for some $\eta = \eta_o > 0$, cannot exit \mathcal{R} through \mathbf{S} for $\eta > \eta_o$.

Proof. We argue by contradiction. Suppose there exists $\eta_1 > \eta_o$ such that $(u(\eta), S_w(\eta), S_g(\eta)) \in \text{int}(\mathcal{R})$ for $\eta < \eta_1$ and $(u(\eta_1), S_w(\eta_1), S_g(\eta_1)) \in \mathbf{S}$. Then at $\eta = \eta_1$ we must have

$$\frac{d}{d\eta} \begin{pmatrix} u \\ S_w \\ S_g \end{pmatrix} \cdot \begin{pmatrix} 0 \\ 1 \\ 1 \end{pmatrix} = (S_w + S_g)' \geq 0 \quad . \quad (4.2)$$

However, considering (TW_+) at η_1 and the fact that $S_w + S_g = f_w + f_g = 1$ we find

$$\begin{aligned} (S_w + S_g)' &= -v - u^+ f_w^+ + v S_w^+ + u^+ \\ &= u^+ f_o^+ - v S_o^+ \\ &= f_o^- - v S_o^- \quad , \end{aligned}$$

where we used $S_g^+ = 0$ and the oil mass balance from (RH) .

We claim that

$$f_o^- - v S_o^- < 0 \quad \text{for all } (S_w^-, S_g^-) \in \ell \quad , \quad (4.3)$$

which would contradict (4.2) and complete the proof. Any point $B \in \ell$ is the end point of a fast rarefaction originating from point A . In terms of the oil saturation this rarefaction satisfies, with $\xi = x/t$,

$$-\xi \frac{dS_o}{d\xi} + \frac{df_o}{d\xi} = 0 \quad \text{for } 0 < \xi < v \quad .$$

Integrating this expression gives

$$-v S_o^- + f_o^- + \int_0^v S_o(\xi) d\xi = 0 \quad , \quad (4.4)$$

which directly implies inequality (4.3). \square

Remark 4.2. Equation (4.4) expresses the oil balance in the steam region. In terms of x and t we have

$$\int_0^{vt} S_o(x, t) dx + (f_o^- - v S_o^-) t = 0 \quad \text{for all } t > 0,$$

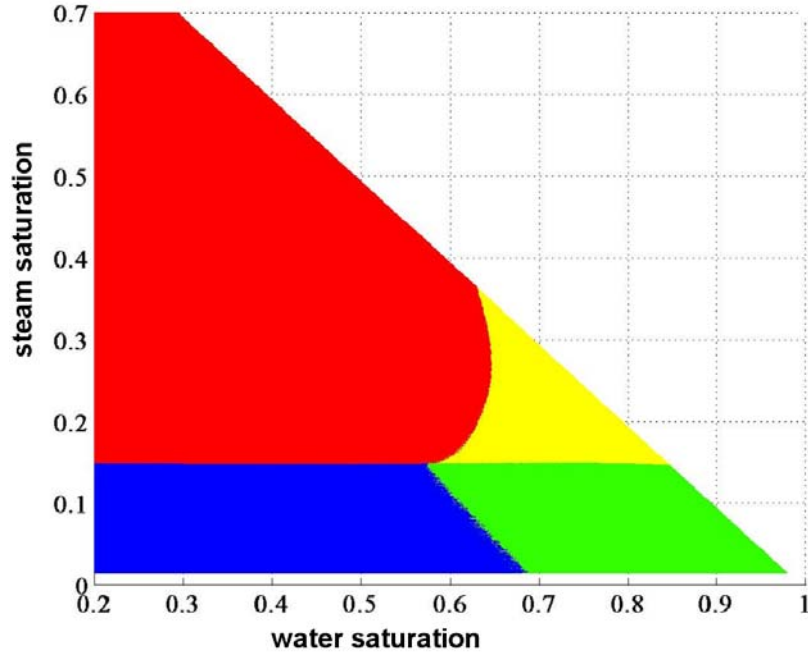
where $(f_o^- - v S_o^-)$ denotes the oil flux with respect to the moving front.

Thus selecting a point on the curve $\Gamma(n)$, the corresponding solution orbit leaves \mathcal{R} through one of the following exit sets (see also Figure 4.3):

$$\begin{aligned} R(\text{red}) &:= \{S_w = 0\} \cup \{u = u^+, 0 < S_w < S_w^+\} \\ B(\text{blue}) &:= \{S_g = 0, 0 < S_w < S_w^+\} \\ Y(\text{yellow}) &:= \{u = u^+, S_w^+ < S_w < 1\} \\ G(\text{green}) &:= \{S_g = 0, S_w^+ < S_w < 1\} \end{aligned}$$

A point on $\Gamma(n)$ is now colored red, blue, yellow or green depending on the exit set of the orbit. For large N and for a large number of points on $\Gamma(n)$, we cover in this way the front triangle below ℓ with these four colors. The point where they meet, denoted by $C = \{u = 1, S_w(C), S_g(C)\}$, and the path through it determines the unique orbit (traveling wave) through \mathcal{R} . For the simplified expressions (3.12) and (3.13) and with

$$v = 0.4, \quad r = 20 \quad \text{and} \quad \alpha = 0.4,$$


 FIGURE 4.4. Color distribution in saturation triangle at \mathbf{T} .

we computed the color distribution. The result is shown in Figure 4.4, where $S_w(C) = 0.582149$ and $S_g(C) = 0.149027$. From the orbits for $\eta < 0$ and $\eta > 0$ we obtain:

$$\begin{aligned} S_w^- &= 0.107351, & S_g^- &= 0.789397, & S_o^- &= 0.103252 \\ S_w^+ &= 0.698487, & S_o^+ &= 0.301513, & u^+ &= 0.610082 \end{aligned}$$

and

$$\Lambda(r) = r \int_0^{\infty} S_g(\eta) d\eta = 0.649858.$$

5. TOWARDS THERMODYNAMIC EQUILIBRIUM

The parameter r in equations (2.20)-(2.22) describes the finite rate condensation process. On physical grounds one expects that the limit $r \rightarrow \infty$ will bring the process in thermodynamic equilibrium with either $T = 1$, meaning boiling point temperature, or $S_g = 0$, with all steam condensing at the steam condensation front. Below we demonstrate this behavior for traveling waves satisfying (TW) , (RH) and (E) . Introducing the notation $S_i = S_i(\eta; r)$ and $u(\eta; r)$, we show that

$$\lim_{r \rightarrow \infty} r(1 - T(\eta)) S_g(\eta; r) = \Lambda(\infty) \delta(\eta), \quad (5.1)$$

where $\Lambda(\infty)$ denotes the limiting-instantaneous condensation rate and δ the Dirac measure at $\eta = 0$. Integrating (5.1) and using (2.25), gives

$$\Lambda(\infty) = \lim_{r \rightarrow \infty} r \int_0^{\infty} S_g(\eta; r) d\eta, \quad (5.2)$$

which is in agreement with condensation rate (3.11) in the hyperbolic limit. Replacing the condensation terms in (3.8) by (5.1), results in the base case discussed in [7].

In terms of the orbit in Figure 4.1, the large r limit means that the point C shifts towards the bottom of \mathcal{R} and that the solution for $\eta > 0$ lies entirely in the bottom set $\{S_g = 0\}$.

To demonstrate (5.1) we integrate the u equation from (TW) in \mathbb{R} . This gives for any $r > 0$

$$\begin{aligned} 1 - u^+(r) &= (1 - \alpha) r \int_{\mathbb{R}} (1 - T(\eta)) S_g(\eta; r) d\eta \\ &= (1 - \alpha) r \int_{\mathbb{R}^+} S_g(\eta; r) d\eta > 0. \end{aligned}$$

Using conditions RH we find

$$\begin{aligned} 1 > u^+(r) &= 1 - (1 - \alpha) (f_g^- - vS_g^-) \\ &> \alpha + (1 - \alpha) vS_g^- > \alpha. \end{aligned}$$

Hence

$$0 < \int_{\mathbb{R}^+} S_g(\eta; r) d\eta < \frac{1}{r} \quad \text{for all } r > 0. \quad (5.3)$$

The S_g -equation in (TW) implies the existence of a constant $\mathcal{L} > 0$ such that

$$|S'_g(\eta; r)| \leq \mathcal{L} \quad \text{for all } \eta, r > 0. \quad (5.4)$$

Hence from (5.3) and (5.4),

$$\lim_{r \rightarrow \infty} S_g(\eta; r) = 0, \quad \text{uniformly in } \eta \geq 0. \quad (5.5)$$

We want to use this convergence in the S_g -equation to control $S'_g(\eta; r)$ as $r \rightarrow \infty$. For this purpose we first write

$$S'_g = uf_g - vS_g - \frac{1}{1 - \alpha} (u - u^+). \quad (5.6)$$

Multiplying this equation by any test function $\varphi \in C_o^\infty(\mathbb{R}^+)$ and integrating the result in \mathbb{R}^+ , gives

$$-\int_{\mathbb{R}^+} S_g \varphi' d\eta = \int_{\mathbb{R}^+} (uf_g - vS_g) \varphi d\eta - \frac{1}{1 - \alpha} \int_{\mathbb{R}^+} (u - u^+) \varphi d\eta.$$

In this expression we send $r \rightarrow \infty$ and use (5.5). This yields

$$\lim_{r \rightarrow \infty} \int_{\mathbb{R}^+} (u(\eta; r) - u^+(r)) \varphi(\eta) d\eta = 0 \quad (5.7)$$

for all $\varphi \in C_o^\infty(\mathbb{R}^+)$.

Once more we consider the u -equation, which we multiply by $\psi \in C_o^\infty(\mathbb{R}^+)$ and integrate in \mathbb{R}^+ to find

$$\int_{\mathbb{R}^+} (u(\eta; r) - u^+(r)) \psi'(\eta) d\eta = (1 - \alpha) r \int_{\mathbb{R}^+} S_g(\eta; r) \psi(\eta) d\eta.$$

Since $\psi' \in C_o^\infty(\mathbb{R}^+)$ as well, we can use (5.7) and obtain

$$\lim_{r \rightarrow \infty} r \int_{\mathbb{R}^+} S_g(\eta; r) \psi(\eta) d\eta = 0 \quad \text{for all } \psi \in C_o^\infty(\mathbb{R}^+), \quad (5.8)$$

which implies

$$\lim_{r \rightarrow \infty} r S_g(\eta; r) = 0, \quad \text{pointwisely in } \eta > 0. \quad (5.9)$$

To see this we use the following argument. For each $n \in \mathbb{N}$, $n > 1$, the interval $(0, \frac{1}{n})$ must contain a point η_n where $r S_g(\eta_n; r) \rightarrow 0$ as $r \rightarrow \infty$. This is a direct consequence of (5.8). Hence for $\eta = \eta_n$ and $\varepsilon > 0$ there exists $r^* > 0$ such that

$$r S_g(\eta_n; r) < \varepsilon \quad \text{for all } r > r^*.$$

Using this and equation (5.6) we have that $S_g(\eta_n; r)$ becomes small with $S_g'(\eta_n; r) < 0$, due to the quadratic terms in f_g , for r sufficiently large. Hence

$$r S_g(\eta; r) < \varepsilon \quad \text{for all } r > r^* \text{ and } \eta \geq \eta_n,$$

implying statement (5.9).

Thus we have shown that

$$\lim_{r \rightarrow \infty} r (1 - T(\eta)) S_g(\eta; r) = 0, \quad \text{pointwisely in } \mathbb{R} \setminus \{0\}$$

and

$$\Lambda(r) = r \int_{\mathbb{R}} (1 - T(\eta)) S_g(\eta; r) d\eta = \frac{1 - u^+(r)}{1 - \alpha} < 1$$

for all $r > 0$. This establishes (5.1) provided

$$\Lambda(r) = \frac{1 - u^+(r)}{1 - \alpha} = (f_g^- - v S_g^-)(r) \quad (\text{from } RH) \quad (5.10)$$

remains strictly positive for all $r > 0$. We verified the behavior of $\Lambda(r)$ numerically. Computational results show that $\Lambda(r)$ depends only slightly on r and changes from $\Lambda(r) = 0.6476$ for $r = 1.95313 \times 10^{-3}$ to $\Lambda(r) = 0.6499$ for $r = 4096$. In the same range S_g^- changes from 0.80993 to 0.789344 (see (5.10)). Note that the four color point C from Figure 4.1, with $S_w = S_w(C)$ and $S_g = S_g(C)$, changes significantly with r . This is shown in Figure 5.1. However, the quantities describing the global solution such as the downstream velocity u^+ and the saturation values S_w^- , S_g^- , and S_w^+ depend only very weakly on r . This is illustrated in in Figure ?? for the upstream saturations.

6. A REALISTIC CASE AND COMPARISON TO PREVIOUS RESULTS

In the previous sections we demonstrate the use of traveling waves to obtain the physically correct saturation and total Darcy velocity values at the steam condensation front. The emphasis was on the principle of construction. Therefore we disregarded various (non-trivial) technicalities, such as temperature dependent viscosities (we still disregard the T dependence of the gas viscosity), and saturation dependent capillary diffusion, coefficients and we disregarded the realistic values of the physical parameters as given in the table.

In this section we are going to carry out the construction in a more realistic setting with the aim to explicit the influence of the finite condensation rate under more practical

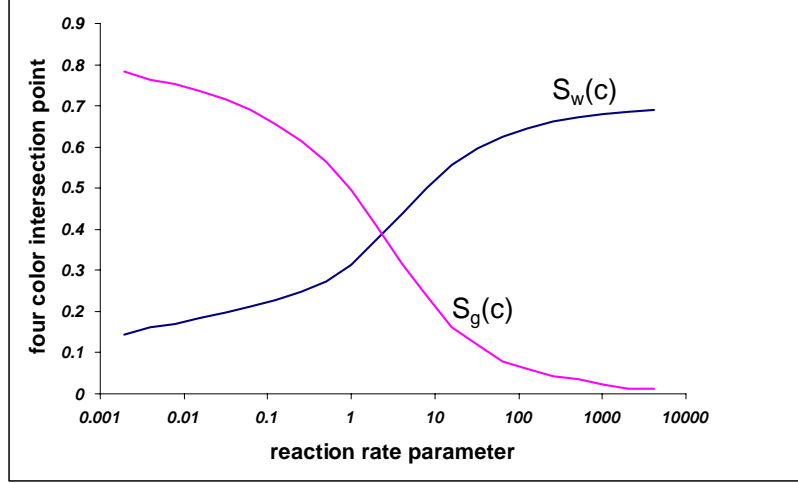


FIGURE 5.1. The four color intersection point C depends strongly on the rate parameter r . The effect on the global solution is not zero, but extremely small.

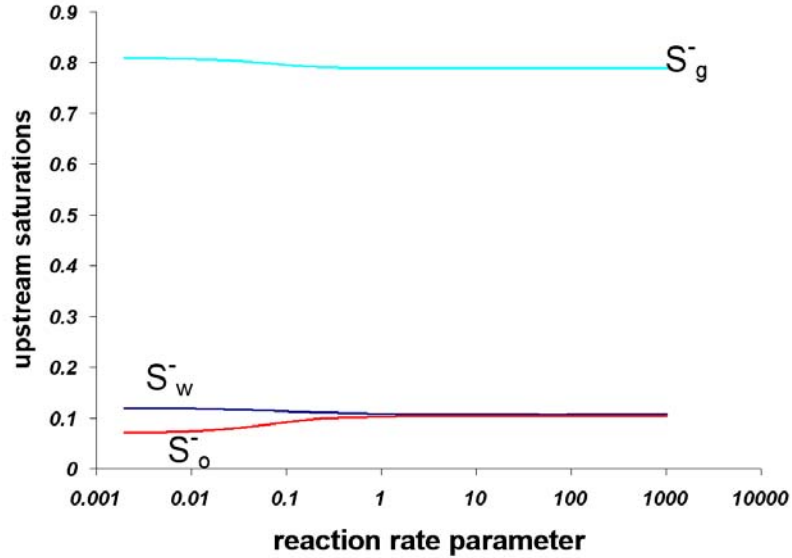


FIGURE 5.2. Upstream saturations depend only weakly on of the reaction rate parameter r .

circumstances. We also give a comparison to the results in [7], where we considered several extensions, but all under equilibrium conditions.

With reference to the table we introduce

- temperature dependent viscosities, yielding the temperature dependent viscosity ratios

$$M_{ow} = \frac{\mu_o}{\mu_w} \quad \text{and} \quad M_{gw} = \frac{\mu_g}{\mu_w};$$

- Brooks-Corey capillary pressures (see [1]), yielding additional terms in the expressions for water and gas discharge (see equations (2.14)-(2.17));

- different relative permeabilities. As in [7] we consider fourth order power law expressions as well as unnormalized Stone I formulas (see [9], [17]):

$$k_{rw} = k_{rw}(S_{we}) = \frac{1}{2} S_{we}^{\frac{2+3\lambda_s}{\lambda_s}} \quad (6.1)$$

$$k_{rg} = k_{rg}(S_{ge}) = (1 - S_{ge})^2 (1 - S_{ge}^{\frac{2+\lambda_s}{\lambda_s}}) \quad (6.2)$$

$$k_{ro} = \frac{kS_o(1 - S_{wc})k_{rg}(S_{we})k_{rw}(S_{ge})}{(1 - S_w)(1 - S_{wc} - S_g)} \quad (6.3)$$

where $S_{we} = \frac{S_w - S_{wc}}{1 - S_{wc}}$, $S_{ge} = \frac{1 - S_g - S_{wc}}{1 - S_{wc}}$. Using a gas saturation dependent residual oil saturation is for the steam drive problem an unnecessary complication and therefore we assume that the residual oil saturation is always zero.

With these extensions the model equations in full dimensional form are

$$\frac{\partial u}{\partial x} = -(1 - \alpha) \frac{q}{\rho_g}, \quad (6.4)$$

$$\varphi \frac{\partial S_w}{\partial t} + \frac{\partial}{\partial x} \left(u f_w - \mathcal{D}_{ww} \frac{\partial S_w}{\partial x} - \mathcal{D}_{wg} \frac{\partial S_g}{\partial x} \right) = \frac{q}{\rho_w}, \quad (6.5)$$

$$\varphi \frac{\partial S_g}{\partial t} + \frac{\partial}{\partial x} \left(u f_g - \mathcal{D}_{gw} \frac{\partial S_w}{\partial x} - \mathcal{D}_{gg} \frac{\partial S_g}{\partial x} \right) = -\frac{q}{\rho_g}. \quad (6.6)$$

Note that the fractional flow functions f_α , and hence the diffusivities $D_{\alpha\beta}$, are discontinuous across the steam condensation front. This is due to the temperature dependence of the mobility ratios.

Using (6.4) to estimate q from (6.5) and (6.6), substituting (2.1) in (6.4) and applying the scalings (2.18) and (2.19) gives

$$\frac{\partial u}{\partial x} = -\frac{q_b L}{u_{inj}} (1 - \alpha) (1 - T) S_g, \quad (6.7)$$

$$\frac{\partial S_w}{\partial t} + \frac{\partial}{\partial x} \left(u f_w - \mathcal{D}^* \left(\mathcal{J}_{ww} \frac{\partial S_w}{\partial x} + \mathcal{J}_{wg} \frac{\partial S_g}{\partial x} \right) \right) = \frac{\alpha}{\alpha - 1} \frac{\partial u}{\partial x}, \quad (6.8)$$

$$\frac{\partial S_w}{\partial t} + \frac{\partial}{\partial x} \left(u f_g - \mathcal{D}^* \frac{\partial}{\partial x} \left(\mathcal{J}_{gw} \frac{\partial S_w}{\partial x} + \mathcal{J}_{gg} \frac{\partial S_g}{\partial x} \right) \right) = -\frac{1}{\alpha - 1} \frac{\partial u}{\partial x}, \quad (6.9)$$

Here

$$\mathcal{D}^* = \frac{\sigma}{\mu_w^* L u_{inj}} \sqrt{k\varphi} \left(\frac{\frac{1}{2} - S_{wc}}{1 - S_{wc}} \right)^{1/\lambda_s}$$

where again μ_w^* is an appropriately chosen characteristic water viscosity (*here the viscosity at the initial reservoir temperature*) and

$$\begin{aligned}
\mathcal{J}_{ww} &= \frac{\mu_w^*}{\mu_w} \frac{\lambda_s f_w}{1 - S_{wc}} \left(\frac{k_{ro}}{M_{ow}} + \frac{k_{rg}}{M_{gw}} \right) \left(\frac{S_w - S_{wc}}{1 - S_{wc}} \right)^{-1/\lambda_s - 1}, \\
\mathcal{J}_{wg} &= -\frac{\mu_w^*}{\mu_w} \frac{\lambda_s f_w}{1 - S_{wc}} \frac{k_{rg}}{M_{gw}} \left(\frac{1 - S_g - S_{wc}}{1 - S_{wc}} \right)^{-1/\lambda_s - 1}, \\
\mathcal{J}_{gw} &= -\frac{\mu_w^*}{\mu_w} \frac{\lambda_s f_g}{1 - S_{wc}} k_{rw} \left(\frac{S_w - S_{wc}}{1 - S_{wc}} \right)^{-1/\lambda_s - 1}, \\
\mathcal{J}_{gg} &= \frac{\mu_w^*}{\mu_w} \frac{\lambda_s f_g}{1 - S_{wc}} \left(\frac{k_{ro}}{M_{ow}} + k_{rw} \right) \left(\frac{1 - S_g - S_{wc}}{1 - S_{wc}} \right)^{-1/\lambda_s - 1}.
\end{aligned} \tag{6.10}$$

Introducing the reciprocal Peclet number ε and the dimensionless rate constant r as

$$\varepsilon := \frac{\sigma}{\mu_w^* L u_{inj}} \sqrt{\mathcal{K}\varphi} \left(\frac{\frac{1}{2} - S_{wc}}{1 - S_{wc}} \right)^{1/\lambda_s}, \tag{6.11}$$

$$r := q_b \frac{\sigma}{\mu_w^* u_{inj}^2} \sqrt{\mathcal{K}\varphi} \left(\frac{\frac{1}{2} - S_{wc}}{1 - S_{wc}} \right)^{1/\lambda_s} = \varepsilon q_b \frac{L}{u_{inj}}, \tag{6.12}$$

applying a traveling wave coordinate transformation $\eta = (x - vt)/\varepsilon$ and integrating the resulting equations leads to the system, with $-\infty < \eta < \infty$,

$$u' = -r(1 - \alpha)(1 - T)S_g, \tag{6.13}$$

$$\mathcal{J}_{ww}S'_w + \mathcal{J}_{wg}S'_g = uf_w - vS_w - \frac{\alpha(1 - u)}{1 - \alpha} - (uf_w - vS_w)^-, \tag{6.14}$$

$$\mathcal{J}_{gw}S'_w + \mathcal{J}_{gg}S'_g = uf_g - vS_g - \frac{u - 1}{1 - \alpha} - (uf_g - vS_g)^-, \tag{6.15}$$

Note that $q_b L/u_{inj}$, the Damkohler number, is considered to be of the same order of magnitude as the Peclet number $1/\varepsilon$. As before, we look for solutions satisfying boundary conditions (BC) subject to the constraints (E) and (RH). Note that equations (6.13)-(6.15) reduce to (TW) when $\mathcal{J}_{wg} = \mathcal{J}_{gw} = 0$ and $\mathcal{J}_{ww} = \mathcal{J}_{gg} = 1$.

Equations (6.14) and (6.15) can be rearranged to explicit expressions for S'_w and S'_g if the determinant

$$\begin{vmatrix} \mathcal{J}_{ww} & \mathcal{J}_{wg} \\ \mathcal{J}_{gw} & \mathcal{J}_{gg} \end{vmatrix} \neq 0.$$

It is straightforward to verify these conditions. Details are omitted. The rearranged equations, with explicit S'_w and S'_g are used in the numerical procedure.

6.1. Procedure for determining the traveling wave orbit. The procedure to find the traveling wave describing the processes in the steam condensation front for the realistic case is slightly different from the procedure used in the previous sections. The reason is that we need a more robust method for solutions with small values of the reaction rate parameter r , for which the four color point (see Figure 4.4) approaches the line ℓ satisfying condition (3.5). Our aim is to find the orbit $D - C - B(n)$ satisfying (6.13)-(6.15), satisfying the Rankine Hugoniot conditions and condition (3.5). First we choose as an initial guess a value n' and determine as in (4.1) the corresponding value of $S_w(n')$. Subsequently we apply condition (3.5)

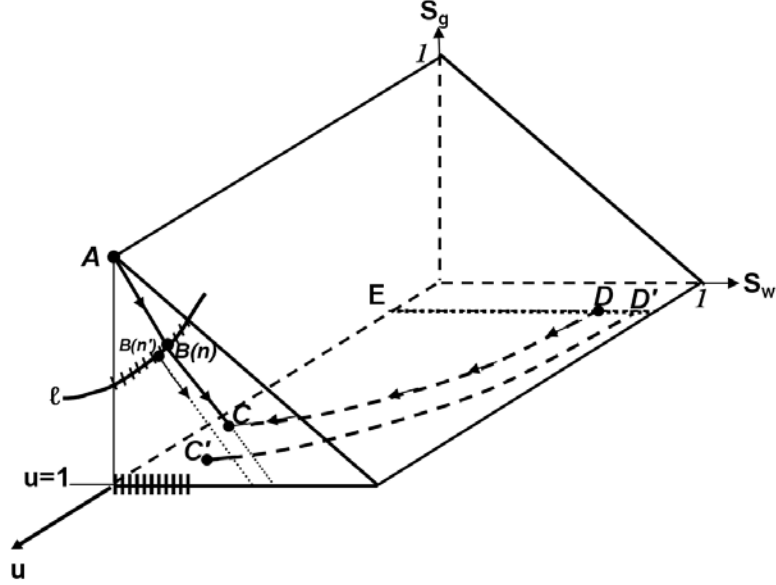


FIGURE 6.1. The orbits from $(u^- = 1, S_g^-, S_w^-)$ in the plane $u = 1$ and orbits in the negative η direction from (u^+, S_g^+, S_w^+) that intersect in the $u = 1$ plane at C are found using bisection (see text).

to determine $S_g(n')$ and thus point $B(n')$ on ℓ (see Figure 6.1). Subsequently we determine from the left (upstream) values of $(u^- = 1, S_w^-, S_g^-)$ at $B(n')$ the right (downstream) values (u^+, S_w^+, S_g^+) i.e. point D' in Figure 6.1, using the Rankine Hugoniot conditions (3.4). At the equilibrium point $B(n')$ we determine the eigenvector to obtain the first point on the orbit away from it. We use this point as initial condition for the rearranged conditions (6.13)-(6.15) in the hot-upstream-region and we determine as before the corresponding orbit until we hit $S_g = 0$. We compute the orbit emanating from $B(n')$ using Eq. (6.13)-(6.15) in the positive η direction until we hit the $S_g = 0$ axis. We call this orbit $\Gamma(n')$. At the downstream point $D' = (u^+, S_w^+, S_g^+)$ we apply a similar procedure i.e. we determine the eigenvector pointing into the domain \mathcal{R} to obtain the first point away from this equilibrium point. Then we solve the rearranged equations (6.13)-(6.15) in the cold-downstream region, this time in the negative η direction until we hit the $u = 1$ plane with values $(S_w(C'), S_g(C'))$ (see Figure 6.1). We choose a second guess e.g. \tilde{n}' to start at a new point $B(\tilde{n}')$ to the right or left with respect to $B(n')$ depending on whether $S_w(C')$ was to the right or to the left of $\Gamma(n')$ and repeat the procedure above. For the sequence of points n', \tilde{n}', \dots we use a bisection routine until we approximate the complete orbit $B(n) - C - D$, with $(S_w(C), S_g(C))$ on the orbit $\Gamma(n)$.

6.2. Results. Figure (6.2) shows the upstream oil saturation as a function of the reaction rate parameter. We distinguish four cases where we use either Stone I expressions (6.1-6.3) (ST) or power law expressions (PL) for the relative permeability and either a constant capillary diffusion (D) (see Table) or a saturation dependent capillary diffusion coefficient (Pc) (see (2.17)). Figure (6.2) also shows, for each of these cases, the oil saturation obtained when thermodynamic equilibrium is assumed [7]. As to be expected from Section 5, the oil

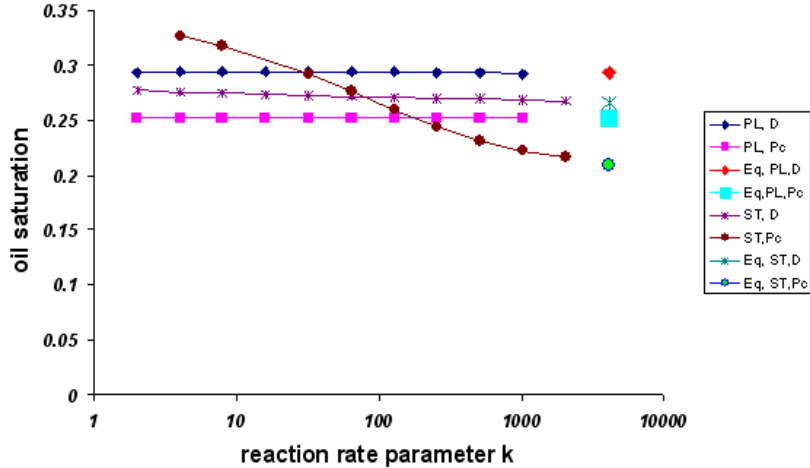


FIGURE 6.2. Upstream oil saturation as a function of the reaction rate parameter. There are four cases. PL = powerlaw relative permeabilities, ST = Stone I relative permeabilities, D = constant capillary diffusion, Pc =saturation dependent capillary diffusion. for constant capillary diffusion (D) or for saturation dependent capillary diffusion coefficients (Pc). The points (Eq, PL, D) , (Eq, PL, Pc) , (Eq, ST, D) and (Eq, PL, Pc) correspond the solutions obtained for thermodynamic equilibrium.

saturation values are about equal to the values obtained for large reaction rate parameters. It is evident that in all cases the global solution depends strongly on the capillary pressure behavior for the same relative permeability expressions. The numerical results, shown in Figure (6.2), suggest that the global solution in terms of the values S_w^-, S_g^-, S_w^+, u^+ is very insensitive to the reaction rate parameter except for the case where we combine saturation dependent capillary pressures and Stone I expressions for the relative permeabilities.

7. CONCLUSIONS

- (1) A hyperbolic model for steam displacement of oil was extended with a finite rate condensation model in the transition zone.
- (2) The traveling wave describing the shock solution is a saddle to saddle connection. Consequently, the global solution depends on the details of the condensation model within the shock.
- (3) Using color coding it has been shown numerically that, given the parameters describing the condensation process, there is a unique set of values S_w^-, S_g^-, S_w^+ and u^+ etc. for which a traveling wave exists (see Figure (4.4)).
- (4) A proof was given that the solution with an infinite reaction rate parameter tends to the solution obtained when thermodynamic equilibrium is assumed.
- (5) The numerical solutions show that there is a dependence of the global solution on the reaction rate parameter. For power law relative permeabilities, this dependence is very weak.
- (6) The procedure described can also be used for a realistic set of input variables. When we combine Stone I relative permeabilities with saturation dependent capillary pressures the effect of the reaction rate parameter is significant.

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