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# On torsion of nonlocal Lam strain gradient FG elastic beams

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### Abstract

The nonlocal strain gradient theory of elasticity is the focus of numerous studies in literature. Eringen's nonlocal integral convolution and Lam's strain gradient model are unified by a variational methodology which leads to well-posed structural problems of technical interest. The proposed nonlocal Lam strain gradient approach is presented for functionally graded (FG) beams under torsion. Static and dynamic responses are shown to be significantly affected by size effects that are assessed in terms of nonlocal and gradient length parameters. Analytical elastic rotations and natural frequencies are established by making recourse to a simple solution procedure which is based on equivalence between integral convolutions and differential equations supplemented with variationally consistent (but non-standard) nonlocal boundary conditions. Effects of Eringen's nonlocal parameter and stretch and rotation gradient parameters on the torsional behavior of FG nano-beams are examined and compared with outcomes in literature. The illustrated methodology is able to efficiently model both stiffening and softening torsional responses of modern composite nano-structures by suitably tuning the small-scale parameters.

## Keywords

Torsion; FG nano-beams; Lam strain gradient elasticity; nonlocal integral elasticity; modified nonlocal strain gradient elasticity; size effects; analytical modelling; NEMS.

## **1. Introduction**

Recent advances in nano-engineering have led to rapid development of smaller and smaller devices in Nanotechnology. Nano-Electro-Mechanical Systems (NEMS) have indeed found a variety of applications in modern nano-systems such as, scanning mirror resonators [1], torsional accelerometer [2], torsional resonator [3], torsional nano-varactor [4], torsional magnetometer [5] and piezoelectric actuators [6]. Mechanical responses of torsional elements of nanoscopic structures should be modelled to achieve optimum design and functionality. However, the well-established approaches of local continuum mechanics are not able to capture size-effects at nano-scales. Nowadays, a variety of higher-order continuum theories, comprising scale parameters, are exploited to model the size-dependency of nano-structures. Characterization and assessment of mechanical responses of nano-structures have attracted enormous attention in the current literature, see e.g. [7-29] and review contributions [30, 31].

In the nonlocal theory of elasticity, nonlocal fields are defined by integral convolutions involving elastic source fields and appropriately selected averaging kernels. Based on the choice of source fields, two basic nonlocal formulations of strain- and stress-driven nonlocal elasticity are considered in literature. The resulted integro-differential equations of the strain-driven Eringen nonlocal integral theory [32] may be replaced with equivalent differential conditions on unbounded domains under the condition of vanishing stress field at infinity. Eringen nonlocal integral model leads to ill-posed structural problems of applicative interest, which are defined in bounded domains, due to incompatibility between constitutive and equilibrium requirements [33-35]. The stress-driven nonlocal formulation, recently conceived in [36], leads instead to well-posed structural problems in nano-engineering. The consistent pure and two-phase stress-driven nonlocal elasticity were successfully exploited to analyze size-dependent elastostatic [37-44] and elastodynamic [45-49] responses of nano-structures.

The strain gradient elasticity theory amounts to a gradient-type material reactive to strain gradients, and consequently, the material behavior of a continuum not only depends on the strain field but also on the gradients of the strain field. The general strain gradient elastic theory was conceived by Mindlin [50] to examine size-dependent responses of elastic materials with micro-structural effects. Advantageously, simplified and modified strain gradient models of elasticity were proposed by Aifantis [51] and Lam et al. [52]. The strain gradient elasticity can be employed to analyze nano-continuu problems, although it is well-known to merely exhibit stiffening structural behaviors.

The nonlocal strain gradient theory of elasticity was introduced by Aifantis [53, 54] as a unified gradient elasticity theory demonstrating the effects of both strain and stress gradients on the constitutive response of elastic materials. The higher-order nonlocal strain gradient theory was then established by Lim et al. [55] by combining the Eringen nonlocal theory and the simplified strain gradient formulation by making recourse to a thermodynamic approach. In nonlocal strain gradient theory, the elasticity form of the stationary Reissner variational principle was proposed [56] and exploited to establish the corresponding beam model [57]. The second-order integro-differential elasticity theory within the thermodynamic framework was also established in [58] introducing the nonlocal effects of higher-order strain gradients. While higher-order boundary conditions associated with the nonlocal strain gradient integral law are not required to be introduced in the analysis of rapidly vanishing fields at infinity, appropriate constitutive boundary conditions should be adopted to close nonlocal strain gradient problems in bounded domains. This objective has been achieved in [59] by detecting appropriate higher-order boundary conditions of constitutive type generating thus the consistent modified nonlocal strain gradient formulation of elasticity. The methodology has been recently utilized to effectively assess size-dependent elasto-static and -dynamic behaviors of structures for nano-engineering applications [60-62].

Structural elements of modern NEMS can often experience torsional deformations, and accordingly, a variety of nonlocal models have been exploited to examine scale phenomena. Instances are as follows: Eringen nonlocal differential model [63-66], Eringen nonlocal viscoelastic model [67, 68], enhanced Eringen nonlocal model [69], strain gradient theory [70-72], stress-driven nonlocal theory [73], mixture stress-driven integral model [74], mixture strain-driven integral model [75] and nonlocal strain gradient theories of elasticity [76, 77]. In the present study, the torsion behavior of functionally graded (FG) elastic nano-beams is analyzed in the novel framework of nonlocal Lam strain gradient elasticity. The proposed formulation can efficiently capture effects of nonlocality of the nano-structure stretch and rotation gradients. Easto-static and -dynamic torsional responses of FG elastic beams are analytically investigated. The outcomes are then compared with the numerical results obtained by the modified nonlocal strain gradient theory. New benchmark examples for nonlocal strain gradient continuum mechanics are also detected that can be advantageously employed for design and optimization of torsional parts of new-generation NEMS.

## 2. Variational elasticity for nonlocal Lam strain gradient FG beams under torsion

Let us consider a straight elastic beam of length L = b - a, having an annular circular crosssection with inner and outer radii  $r_i$  and  $r_o$  occupying a domain  $\Xi$ , as schematically depicted in Fig. 1. The abscissa x is selected along the beam axis and the axes (y,z) represent cartesian components of the position vector **r** of a cross-sectional point with respect to the centroid. The FG nano-beam is considered to be made of two different elastically homogeneous materials with densities  $\rho_i, \rho_o$  and shear moduli  $G_i, G_o$ , respectively at the inner and outer surfaces. Thus, the effective material properties of the FG nano-beam continuously vary along the radial direction **r** in the beam cross-section plane. Shear modulus and material density are therefore assumed to vary along the radial direction according to the general form

$$G = G(\mathbf{r}), \qquad \rho = \rho(\mathbf{r}) \tag{1}$$

To examine the size-dependent behaviour of FG elastic nano-beams with continuous radial variation of material properties, it is convenient to introduce the effective local elastic shear stiffness  $A_G$  and effective torsional stiffness  $J_G$  as

$$A_{G} = \iint_{\Xi} G(\mathbf{r}) dA, \qquad J_{G} = \iint_{\Xi} G(\mathbf{r}) (\mathbf{r} \Box \mathbf{r}) dA \qquad (2)$$

with the dot into the integrals standing for inner product between vectors. Likewise, the effective mass polar moment of inertia  $J_{\rho}$ , to be employed in the elasto-dynamic torsional analysis of FG nano-beam, is introduced as

$$J_{\rho} = \iint_{\Xi} \rho(\mathbf{r})(\mathbf{r} \Box \mathbf{r}) dA$$
(3)

A detailed discussion on elastic properties of FG cross-sections in the torsion analysis is provided in Ref. [67].



Fig. 1. Coordinate system and configuration of a FG beam under torsion

Cartesian components of the displacement field of the beam under torsion, up to an inessential additional rigid body motion, take the form

$$u_x = 0, \qquad u_y = -\varphi(x)z, \qquad u_z = \varphi(x)y \qquad (4)$$

with  $\varphi:[a,b] \mapsto \Re$  standing for torsional rotation.

The non-vanishing kinematically compatible shear strain field is accordingly provided by

$$\boldsymbol{\gamma} = \mathbf{Rr}\kappa(x,t) = \mathbf{Rr}\partial_x \varphi(x,t)$$
(5)

where  $\kappa = \partial_x \varphi : [a,b] \mapsto \Re$  is the geometric torsional curvature along with the shear strain vector  $\gamma$ , shear stress vector  $\tau$ , position vector  $\mathbf{r}$  and rotation tensor  $\mathbf{R}$  expressed by

$$\boldsymbol{\gamma} = \begin{bmatrix} \gamma_{yx} \\ \gamma_{zx} \end{bmatrix}, \qquad \boldsymbol{\tau} = \begin{bmatrix} \tau_{yx} \\ \tau_{zx} \end{bmatrix}, \qquad \boldsymbol{r} = \begin{bmatrix} y \\ z \end{bmatrix}, \qquad \boldsymbol{R} = \begin{bmatrix} 0 & -1 \\ 1 & 0 \end{bmatrix}$$
(6)

The vector **Rr** provides the  $\pi/2$  counterclockwise rotation of position vector **r**.

The beam is assumed to be subjected to distributed torsional couples per unit-length  $q_t : [a,b] \mapsto \Re$  and concentrated couples at the end cross-sections  $\overline{T_a}$  and  $\overline{T_b}$ . Standardly, the differential and boundary conditions of equilibrium write as

$$\partial_{x}T + q_{t} = J_{\rho}\partial_{tt}\varphi$$

$$\left(T + \overline{T_{a}}\right)\delta\varphi\Big|_{x=a} = \left(T - \overline{T_{b}}\right)\delta\varphi\Big|_{x=b} = 0$$
(7)

where the twisting resultant moment is denoted by T and provided by

$$T = \iint_{\Xi} (\mathbf{Rr}) \Box \tau dA \tag{8}$$

The total potential energy *U*, consistent with modified strain gradient theory of Lam et al. [52], depends on the torsional curvature  $\kappa \in C^2([a,b];\Re)$  and may be shown to be [70]

$$U(\kappa) \coloneqq \frac{1}{2} \int_{a}^{b} \left( \left( J_{G} + 3A_{G}\ell_{2}^{2} \right) \kappa^{2} + J_{G} \left( \frac{8}{15} \ell_{1}^{2} + \frac{1}{4} \ell_{2}^{2} \right) \left( \partial_{x} \kappa \right)^{2} \right) dx$$

$$\tag{9}$$

where  $\ell_1$  and  $\ell_2$  correspondingly designate the gradient characteristic parameters associated with the nano-structural stretch and rotation gradients. The effective local elastic shear stiffness  $A_G$  and effective torsional stiffness  $J_G$  are also given by Eq. (2).

Motivated by the milestone approach of Eringen [32], the nonlocal Lam strain gradient model of torsion problem can be formulated by including nonlocal integral convolution in Eq. (9). The total elastic strain energy of nonlocal Lam strain gradient FG nano-beams under torsion  $\Pi$  is

$$\Pi(\kappa) \coloneqq \frac{1}{2} \int_{a}^{b} \left( \left( J_{G} + 3A_{G}\ell_{2}^{2} \right) \left( \psi_{\lambda} \ast \kappa \right) \kappa + J_{G} \left( \frac{8}{15} \ell_{1}^{2} + \frac{1}{4} \ell_{2}^{2} \right) \left( \psi_{\lambda} \ast \partial_{x} \kappa \right) \left( \partial_{x} \kappa \right) \right) dx$$

$$(10)$$

where the integral convolution of a scalar field f with a smoothing kernel  $\psi_{\lambda}$  is defined by

$$(\psi_{\lambda} * f)(x) \coloneqq \int_{a}^{b} \psi_{\lambda}(x - \overline{x}) f(\overline{x}) d\overline{x}$$
(11)

The smoothing kernel $\psi_{\lambda}$  is well-established to fulfil the positivity and parity, symmetry, normalization and limit impulsivity properties [33-35].

The twisting moment in the nonlocal Lam strain gradient model  $T \in C^1([a,b]; \mathfrak{R})$  is established via a mathematically well-posed variational constitutive condition as follows

$$\langle T, \delta \kappa \rangle \coloneqq \int_{a}^{b} T(x) \delta \kappa(x) dx = \langle d \Pi(\kappa), \delta \kappa \rangle$$
 (12)

for any virtual torsional curvature field  $\delta \kappa \in C_0^1([a,b]; \Re)$  having compact support in [a,b]. The directional derivative of the elastic strain energy along a virtual torsional curvature can be determined via introducing the expression of  $\Pi$  as Eq. (10) based on integration by parts

$$\left\langle d\Pi(\kappa), \delta\kappa \right\rangle = \int_{a}^{b} \left( \left( J_{G} + 3A_{G}\ell_{2}^{2} \right) (\psi_{\lambda} * \kappa) \delta\kappa + J_{G} \left( \frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2} \right) (\psi_{\lambda} * \partial_{x}\kappa) \partial_{x} \left( \delta\kappa \right) \right) dx$$

$$= \int_{a}^{b} \left( \left( J_{G} + 3A_{G}\ell_{2}^{2} \right) (\psi_{\lambda} * \kappa) - J_{G} \left( \frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2} \right) \partial_{x} \left( \psi_{\lambda} * \partial_{x}\kappa \right) \right) \delta\kappa dx$$

$$+ J_{G} \left( \frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2} \right) \left( (\psi_{\lambda} * \partial_{x}\kappa) \delta\kappa \Big|_{x=b} - (\psi_{\lambda} * \partial_{x}\kappa) \delta\kappa \Big|_{x=a} \right)$$

$$(13)$$

The test fields  $\delta \kappa \in C_0^1([a,b]; \Re)$  in the variational condition Eq. (12) have compact supports, so that  $\delta \kappa|_{x=a} = \delta \kappa|_{x=b} = 0$ , and thus the boundary terms in Eq. (13) are disappearing.

A standard localization procedure provides the nonlocal strain gradient twisting moment *T* in terms of the torsional curvature field  $\kappa$  while imposing the variational condition Eq. (12)

$$T(x) = \left(J_G + 3A_G\ell_2^2\right)\left(\psi_\lambda *\kappa\right)\left(x\right) - J_G\left(\frac{8}{15}\ell_1^2 + \frac{1}{4}\ell_2^2\right)\partial_x\left(\psi_\lambda *\partial_x\kappa\right)\left(x\right)$$
(14)

The equivalent differential constitutive problem with the corresponding constitutive boundary conditions associated with the nonlocal Lam strain gradient model (NLSG) are determined following the mathematically well-posed approach by Barretta and Marotti de Sciarra [59]. The smoothing kernel $\psi_{\lambda}$  in the nonlocal integral convolution Eq. (14) is assumed to be the Helmholtz bi-exponential function considered in Eringen nonlocal theory

$$\psi_{\lambda}(x) \coloneqq \frac{1}{2L_{c}} \exp\left(-\frac{|x|}{L_{c}}\right)$$
(15)

where the nonlocal characteristic length  $L_c = \lambda L$  describes long-range interactions.

**Proposition.** The nonlocal strain gradient constitutive law Eq. (14), endowed with the Helmholtz bi-exponential kernel Eq. (15), is equivalent to the differential constitutive law

$$T(x) - L_c^2 \partial_{xx} T(x) = \left(J_G + 3A_G \ell_2^2\right) \kappa(x) - J_G\left(\frac{8}{15}\ell_1^2 + \frac{1}{4}\ell_2^2\right) \partial_{xx} \kappa(x)$$

$$\tag{16}$$

equipped with the constitutive boundary conditions (CBC)

$$\partial_{x}T(a) - \frac{1}{L_{c}}T(a) = \frac{1}{L_{c}^{2}}J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\partial_{x}\kappa(a)$$

$$\partial_{x}T(b) + \frac{1}{L_{c}}T(b) = \frac{1}{L_{c}^{2}}J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\partial_{x}\kappa(b)$$
(17)

The introduced non-standard boundary conditions Eq. (17) are of constitutive type, naturally associated with the nonlocal Lam strain gradient law Eq. (16) for twisted FG elastic nanobeams. Noteworthy, the constitutive boundary conditions do not contradict equilibrium requirements and thus result in mathematically well-posed size-dependent problems.

As the nonlocal characteristic length tends to zero, the Lam strain gradient model of straight beams under torsion can be recovered as a particular limiting case of the nonlocal Lam strain gradient law. Due to the limit impulsivity property of kernel $\psi_{\lambda}$ , nonlocal strain gradient constitutive law Eq. (14) can be simplified to the Lam strain gradient differential equation

$$T(x) = \left(J_G + 3A_G\ell_2^2\right)\kappa(x) - J_G\left(\frac{8}{15}\ell_1^2 + \frac{1}{4}\ell_2^2\right)\partial_{xx}\kappa(x)$$
(18)

equipped with higher-order boundary conditions determined via simplifying NLSG CBC Eq. (17) on beam ends  $\partial[a,b]$  as

$$J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\partial_{x}\kappa(a) = 0$$

$$J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\partial_{x}\kappa(b) = 0$$
(19)

The gradient constitutive law Eq. (18) and the associated non-standard boundary conditions Eq. (19) can be also detected by a formal application of Hamilton's principle [70].

Nonlocal strain gradient model suffers from prescribing un-motivated higher-order boundary conditions of strain gradient theory. On the contrary, the consistent form of constitutive boundary conditions suitably associated with the nonlocal strain gradient constitutive law is recently introduced in the framework of modified nonlocal strain gradient model [59].

Modified nonlocal strain gradient model (MNSG) leads to well-posed problems, and thus, it is selected here for comparison sake. The formulation of MNSG is briefly recalled as follows.

MNSG model of FG elastic beam under torsion is formulated by expressing the twisting resultant moment *T* in terms of two integral convolutions guided by elastic torsional curvature  $\kappa$  and by its gradient along the beam axis  $\partial_x \kappa$  as

$$T(x) = J_G(\psi_{\lambda} * \kappa)(x) - J_G \ell_s^2 \partial_x (\psi_{\lambda} * \partial_x \kappa)(x)$$

where gradient characteristic length  $\ell_s$  is introduced in the MNSG to make dimensionally homogeneous the convolutions in Eq. (20). Following the seminal contribution by Barretta and Marotti de Sciarra [59], it may be shown that the nonlocal strain gradient constitutive law Eq. (20) equipped with the Helmholtz bi-exponential kernel Eq. (15) is equivalent to the differential constitutive relation

$$T(x) - L_c^2 \partial_{xx} T(x) = J_G \kappa(x) - J_G \ell_s^2 \partial_{xx} \kappa(x)$$
(21)

equipped with the constitutive boundary conditions at the beam ends  $\partial[a,b]$ 

$$\partial_{x}T(a) - \frac{1}{L_{c}}T(a) = \frac{\ell_{s}^{2}}{L_{c}^{2}}J_{G}\partial_{x}\kappa(a)$$

$$\partial_{x}T(b) + \frac{1}{L_{c}}T(b) = \frac{\ell_{s}^{2}}{L_{c}^{2}}J_{G}\partial_{x}\kappa(b)$$
(22)

Notwithstanding that the constitutive boundary conditions are naturally associated with the nonlocal strain gradient constitutive law, higher-order boundary conditions of strain gradient theory as Eq. (19) are improperly adopted in the literature [76, 77].

## 3. Elastostatic torsion

Differences in the elastostatic torsional behavior of FG nano-beams are demonstrated adopting the nonlocal Lam strain gradient theory and the modified nonlocal strain gradient model to doubly clamped and cantilever nano-beams subjected to a uniformly distributed couples  $\bar{q}_r$ . To appropriately compare the size-dependent behavior of twisted FG elastic nano-beams, the effects corresponding to each gradient characteristic parameter in the NLSG are independently examined and compared with the counterpart results of MNSG. The nondimensional parameters: axial abscissa  $\bar{x}$ , nonlocal characteristic parameter  $\lambda$ , gradient characteristic parameter  $\mu$ , torsional rotation  $\bar{\varphi}$  as well as the radius of gyration  $\bar{r}$  are defined by

$$\bar{x} = \frac{x}{L}, \qquad \lambda = \frac{L_c}{L}, \qquad \mu = \frac{\ell}{L}, \qquad \bar{\varphi}(\bar{x}) = \varphi(x) \frac{J_G}{\bar{q}_t L^2}, \qquad \bar{r} = \frac{1}{L} \sqrt{\frac{J_G}{A_G}}$$
(23)

In the elastostatic analysis, the inertia term in the differential condition of equilibrium Eq.  $(7)_1$  vanishes and the equilibrated twisting moment *T* has thus the expression

$$T(x) = -\int_{a}^{x} q_{t}(\zeta) d\zeta + \Upsilon_{1}$$
(24)

up to the integration constant  $\Upsilon_1$ . The torsional curvature  $\kappa$  is detected as a result of solving the constitutive differential equation of NLSG Eq. (16) or MNSG Eq. (21) in terms of integration constants  $\Upsilon_2, \Upsilon_3$ 

$$\kappa(x) = \Upsilon_{2} \exp\left(-x\sqrt{\frac{\alpha}{\beta}}\right) + \Upsilon_{3} \exp\left(x\sqrt{\frac{\alpha}{\beta}}\right) + \frac{1}{2\sqrt{\alpha\beta}} \exp\left(-x\sqrt{\frac{\alpha}{\beta}}\right) \int_{a}^{x} \exp\left(\xi\sqrt{\frac{\alpha}{\beta}}\right) (T(\xi) - L_{c}^{2}\partial_{\xi\xi}T(\xi)) d\xi - \frac{1}{2\sqrt{\alpha\beta}} \exp\left(x\sqrt{\frac{\alpha}{\beta}}\right) \int_{a}^{x} \exp\left(-\eta\sqrt{\frac{\alpha}{\beta}}\right) (T(\eta) - L_{c}^{2}\partial_{\eta\eta}T(\eta)) d\eta$$
(25)

where  $\alpha, \beta$  for NLSG and MNSG are defined as

$$\alpha \big|_{\text{NLSG}} = J_G + 3A_G \ell_2^2, \qquad \alpha \big|_{\text{MNSG}} = J_G$$

$$\beta \big|_{\text{NLSG}} = J_G \left(\frac{8}{15}\ell_1^2 + \frac{1}{4}\ell_2^2\right), \qquad \beta \big|_{\text{MNSG}} = J_G \ell_s^2 \qquad (26)$$

The differential condition of kinematic compatibility  $\kappa = \partial_x \varphi$  has to be subsequently integrated to detect the torsional rotation  $\varphi$  up to the integration constant  $\Upsilon_4$ 

$$\varphi(x) = \int_{a}^{x} \kappa(\zeta) d\zeta + \Upsilon_{4}$$
(27)

The integration constants  $\Upsilon_k$  (k = 1..4) can be evaluated by imposing two standard kinematic and static boundary conditions (BC) as well as the corresponding two CBCs associated with NLSG or MNSG. The proposed solution technique leads to exact analytical solutions by integrating differential equations of lower order. The acronyms LOC, NLSG and MNSG stand for local beam model, nonlocal Lam strain gradient model and modified nonlocal strain gradient model, respectively.

## 3.1. Cantilever twisted FG nano-beam subjected to uniform couples

The classical BC for a cantilever FG elastic nano-beam subjected to uniform couples write as

$$\varphi(a) = 0, \qquad T(b) = 0 \tag{28}$$

The non-dimensional torsional rotation of the cantilever beam is determined by applying the proposed solution technique while imposing the classical BCs and the corresponding CBCs. It may be shown that the maximum value of the torsional rotation of NLSG and MNSG is

$$\overline{\varphi}_{\max}^{\text{NLSG}} = \left(\frac{1}{2} + \lambda\right) \frac{\overline{r}^2}{\overline{r}^2 + 3\mu_2^2}$$

$$\overline{\varphi}_{\max}^{\text{MNSG}} = \frac{1}{2} + \lambda$$
(29)

which means that the torsional rotation of the FG elastic nano-beam at the free end is independent of the gradient characteristic parameters  $\mu_1$  and  $\mu_s$  for NLSG and MNSG, respectively. Accordingly, the value of the torsional rotation at the mid-span of the cantilever FG elastic beam is examined to analyze the effects of nonlocal and gradient characteristic parameters

$$\overline{\varphi}^{\text{NLSG}}\Big|_{\overline{x}=\frac{1}{2}} = \frac{\overline{r}^{2}}{120(\overline{r}^{2}+3\mu_{2}^{2})^{2}} \left(45\overline{r}^{2}+120\lambda\overline{r}^{2}+120\lambda^{2}\overline{r}^{2}-64\overline{r}^{2}\mu_{1}^{2}+135\mu_{2}^{2}+360\lambda\mu_{2}^{2}+360\lambda^{2}\mu_{2}^{2}\right) \\ -30\overline{r}^{2}\mu_{2}^{2}-2\operatorname{sech}\left(\frac{1}{\overline{r}}\sqrt{\frac{15(\overline{r}^{2}+3\mu_{2}^{2})}{32\mu_{1}^{2}+15\mu_{2}^{2}}}\right) \left(30\lambda(1+2\lambda)(\overline{r}^{2}+3\mu_{2}^{2})-\overline{r}^{2}\left(32\mu_{1}^{2}+15\mu_{2}^{2}\right)\right)\right)$$
(30)  
$$\overline{\varphi}^{\text{MNSG}}\Big|_{\overline{x}=\frac{1}{2}} = \frac{1}{8} \left(3+8\lambda(1+\lambda)-8\mu_{s}^{2}-4(\lambda+2\lambda^{2}-2\mu_{s}^{2})\operatorname{sech}\left(\frac{1}{2\mu_{s}}\right)\right)$$

# 3.2. Doubly clamped FG nano-beam subjected to uniform couples

For a FG elastic nano-beam subjected to uniform couples with doubly clamped ends, the kinematic BCs are given by

$$\varphi(a) = 0, \qquad \qquad \varphi(b) = 0 \tag{31}$$

The non-dimensional torsional rotation can be detected exploiting the proposed solution methodology and prescribing the kinematic BCs and the CBCs. For numerical illustrations, the maximum non-dimensional torsional rotation is determined as

$$\overline{\varphi}_{\max}^{\text{NLSG}} = \frac{\overline{r}^2}{120(\overline{r}^2 + 3\mu_2^2)^2} \left( 15\overline{r}^2 + 60\lambda\overline{r}^2 + 120\lambda^2\overline{r}^2 - 64\overline{r}^2\mu_1^2 + 45\mu_2^2 + 180\lambda\mu_2^2 + 360\lambda^2\mu_2^2 - 30\overline{r}^2\mu_2^2 - 2\operatorname{sech}\left(\frac{1}{\overline{r}}\sqrt{\frac{15(\overline{r}^2 + 3\mu_2^2)}{32\mu_1^2 + 15\mu_2^2}}\right) \left( 30\lambda(1 + 2\lambda)(\overline{r}^2 + 3\mu_2^2) - \overline{r}^2(32\mu_1^2 + 15\mu_2^2)) \right) \right)$$
(32)  
$$\overline{\varphi}_{\max}^{\text{MNSG}} = \frac{1}{8} \left( 1 + 4\lambda + 8\lambda^2 - 8\mu_s^2 - 4(\lambda + 2\lambda^2 - 2\mu_s^2) \operatorname{sech}\left(\frac{1}{2\mu_s}\right) \right)$$

## 3.3. Numerical results of elastostatic torsion

The normalized torsional rotations at the mid-span of the FG elastic nano-beams associated with the nonlocal Lam strain gradient model are illustrated in Figs. 2-5 for cantilever and doubly clamped beams subjected to uniformly distributed couples. The effects of stretch and rotation gradient parameters in the framework of nonlocal Lam strain gradient model are separately examined and the results consistent with the NLSG with  $\ell_1 \neq 0$  are compared with the counterpart results of the modified nonlocal strain gradient model. The demonstrated torsional rotation  $\overline{\phi}$  is normalized with respect to the corresponding torsional rotation of the local beam model  $\bar{\varphi}^{\text{LOC}}$ . In Figs. 2-3 where 2D comparison is made between the torsional response of NLSG (with  $\ell_1 \neq 0$ ) and MNSG beams, the nonlocal characteristic parameter  $\lambda$  is ranging in the interval ]0,1[. The stretch gradient parameter of NLSG  $\mu_1$  and gradient characteristic parameter of MNSG  $\mu_s$  are ranging in the set of {0.1, 0.3, 0.5, 0.7, 1.0} while the rotation gradient parameter of NLSG  $\mu_2$  is assumed to vanish. Figs. 4 and 5 demonstrate the 3D variations of normalized torsional rotation at the mid-span of the FG elastic nano-beam versus the nonlocal characteristic parameters where the effects of stretch and rotation gradient parameters are independently investigated. All the characteristic parameters in Figs. 4 and 5 are ranging in the interval ]0,1[. Furthermore, in all of the illustrated results associated with NLSG, the non-dimensional radius of gyration is assumed as  $\overline{r} = 1/10\sqrt{2}$ .

It is noticeably deduced from Figs. 2-5 that the adopted size-dependent NLSG and MNSG models exhibit a softening behavior in terms of the nonlocal characteristic parameter  $\lambda$ , that is a larger  $\lambda$  involves a larger torsional rotation for a given gradient characteristic parameter. The torsional rotation of FG elastic nano-beam also decreases as the gradient parameters increase, and accordingly, both nonlocal strain gradient theories demonstrate a stiffening behavior in terms of the gradient parameters for a given value of  $\lambda$ . Notably, the torsional

response of nano-beams associated with both NLSG and MNSG models are in full agreement as the gradient characteristic parameters tend to zero. While the torsional response of nanobeam associated with the MNSG underestimates the counterpart torsional rotation of NLSG beam with non-vanishing stretch gradient parameter  $\mu_1$ , it is strictly higher than the torsional rotation of NLSG beam with non-vanishing rotation gradient parameter  $\mu_2$ . This is clearly due to the dominant stiffening effects of rotation gradient parameter  $\mu_2$  in comparison to stretch gradient parameter  $\mu_1$  in NLSG. As expected, the size-dependent elastic torsional rotation of nano-beams in accordance with either of the nonlocal strain gradient models coincides with the local response for vanishing nonlocal and gradient parameters.

The numerical values of normalized maximum and mid-span torsional rotation of doubly clamped and cantilever FG elastic nano-beams examined in the framework of NLSG and MNSG are, respectively, collected in Tables 1-2 and 3-4.



Fig. 2. Cantilever FG nano-beams under uniform couples: normalized mid-span torsional rotation  $\overline{\varphi}_{(x=1/2)}$  versus nonlocal and gradient parameters by NLSG  $(\ell_1 \neq 0)$  and MNSG



Fig. 3. Doubly clamped FG nano-beams under uniform couples: normalized maximum torsional rotation  $\overline{\varphi}_{max}$  versus nonlocal and gradient parameters by NLSG  $(\ell_1 \neq 0)$  and

MNSG



**Fig. 4.** Uniformly loaded cantilever FG nano-beams: effects of  $\lambda, \mu$  on normalized  $\overline{\varphi}_{(x=1/2)}$  by



NLSG  $(\ell_1 \neq 0)$ , NLSG  $(\ell_2 \neq 0)$  and MNSG

Fig. 5. Uniformly loaded doubly clamped FG nano-beams: effects of  $\lambda$ ,  $\mu$  on normalized

$$\bar{\varphi}_{max}$$
 by NLSG  $(\ell_1 \neq 0)$ , NLSG  $(\ell_2 \neq 0)$  and MNSG

## 4. Torsional elastic vibrations

## 4.1. Free vibration analysis

To examine the torsional free vibrations of FG elastic nano-beams, applied distributed torsional couples are assumed to vanish in the differential condition of equilibrium. The twisting moment *T* is detected by applying the differential condition of equilibrium Eq.  $(7)_1$  to the constitutive differential law of NLSG Eq. (16) as

$$T = J_{\rho}L_{c}^{2}\partial_{xtt}\varphi + \left(J_{G} + 3A_{G}\ell_{2}^{2}\right)\kappa - J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\partial_{xx}\kappa$$

$$(33)$$

Employing the kinematic compatibility  $\kappa = \partial_x \varphi$ , the differential condition of dynamic equilibrium governing the torsional vibrations is consequently given by

$$J_{\rho}\partial_{tt}\varphi - J_{\rho}L_{c}^{2}\partial_{xxtt}\varphi = \left(J_{G} + 3A_{G}\ell_{2}^{2}\right)\partial_{xx}\varphi - J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\partial_{xxxx}\varphi$$
(34)

equipped with the classical boundary conditions Eq.  $(7)_2$  and corresponding constitutive boundary conditions of NLSG Eq. (17). A well-established procedure of separating spatial and time variables is then utilized to analyze the relevant vibrational problem

$$\varphi(x,t) = \Phi(x) \exp(i \, \omega t) \tag{35}$$

where  $i = \sqrt{-1}$ ,  $\Phi$  and  $\omega$  are the torsional mode shapes and natural frequency of vibrations. The governing equation of the torsional coordinate function  $\Phi$  can be obtained by enforcing the separation of variables Eq. (35) to the differential condition of dynamic equilibrium Eq. (34)

$$J_{G}\left(\frac{8}{15}\ell_{1}^{2} + \frac{1}{4}\ell_{2}^{2}\right)\frac{d^{4}\Phi}{dx^{4}} + \left(J_{\rho}L_{c}^{2}\omega^{2} - \left(J_{G} + 3A_{G}\ell_{2}^{2}\right)\right)\frac{d^{2}\Phi}{dx^{2}} - J_{\rho}\omega^{2}\Phi = 0$$
(36)

The torsional coordinate function can be analytically detected as

$$\Phi(x) = \Gamma_1 \sin\zeta_1 x + \Gamma_2 \cos\zeta_1 x + \Gamma_3 \sinh\zeta_2 x + \Gamma_4 \cosh\zeta_2 x$$
(37)

where the unknown constants  $\Gamma_k (k = 1..4)$  are determined by prescribing suitable classical and constitutive boundary conditions, along with

$$\zeta_{1}^{2} = \frac{\left(J_{\rho}L_{c}^{2}\omega^{2} - \alpha\right) + \sqrt{\left(J_{\rho}L_{c}^{2}\omega^{2} - \alpha\right)^{2} + 4J_{\rho}\omega^{2}\beta}}{2\beta}}{2\beta}$$

$$\zeta_{2}^{2} = \frac{-\left(J_{\rho}L_{c}^{2}\omega^{2} - \alpha\right) + \sqrt{\left(J_{\rho}L_{c}^{2}\omega^{2} - \alpha\right)^{2} + 4J_{\rho}\omega^{2}\beta}}{2\beta}}{2\beta}$$
(38)

with  $\alpha, \beta$  introduced for NLSG and MNSG as Eq. (26).

To illustrate the solution procedure for detecting the torsional natural frequency of FG elastic nano-beam, let us consider a cantilever FG nano-beam in the nonlocal Lam strain gradient theory. A homogeneous fourth-order algebraic system in terms of the unknown constants  $\Gamma_k$  (k = 1..4) is formulated by prescribing the classical BC Eq. (28) along with the CBC Eq. (17) in the torsional coordinate function Eq. (37). Similarly, homogeneous fourth-order algebraic systems can be detected for nano-beams associated with either of nonlocal strain gradient models of NLSG or MNSG. To detect the non-trivial solution of torsional free vibrations, the system of algebraic equations has to be singular, that is the determinant of the coefficients of the homogeneous system has to vanish. For cantilever and doubly clamped FG elastic beams consistent with either of the nonlocal strain gradient models, a highly nonlinear characteristic equation in terms of torsional natural frequencies is found that is numerically solved.

## 4.2. Numerical results of torsional free vibrations

Torsional natural frequency of cantilever and doubly clamped FG elastic nano-beams in the framework of nonlocal Lam strain gradient theory and modified nonlocal strain gradient model are numerically evaluated here. The effects of stretch and rotation gradient parameters associated with NLSG are separately examined and the results consistent with NLSG with  $\ell_1 \neq 0$  are compared with the counterpart results of the MNSG. For consistency, the non-dimensional torsional fundamental frequency  $\overline{\omega}$  is introduced in the illustrations as

$$\overline{\omega}^2 = \left(\frac{L^2 J_{\rho}}{\pi^2 J_G}\right) \omega^2 \tag{39}$$

The illustrated torsional fundamental frequencies are furthermore normalized employing their corresponding local counterparts  $\bar{\omega}_{LOC}$ .

The effects of characteristic parameters on the normalized fundamental frequency of the nonlocal Lam strain gradient model with non-vanishing stretch gradient parameters  $\ell_1 \neq 0$  in comparison with the counterpart results of the modified nonlocal strain gradient model are exhibited in Figs. 6 and 7 for cantilever and doubly clamped FG nano-beams. The variations of the normalized torsional frequencies associated with the nonlocal Lam strain gradient model with non-vanishing rotation gradient parameters  $\ell_2 \neq 0$  are depicted in Figs. 8 and 9. The nonlocal and gradient characteristic parameters in Figs. 6 and 7 are considered to range in the same set as the elastostatic torsional response illustrated in Figs. 2-3. Also in the results consistent with NLSG with  $\ell_2 \neq 0$ , as illustrated in Fig. 8 and 9, the rotation gradient parameter  $\mu_2$  is ranging in the set of  $\{0.01, 0.1, 0.3, 0.5, 0.7, 1.0\}$ , the nonlocal parameter  $\lambda$  is ranging in the interval [0,1] and the stretch gradient parameter  $\mu_1$  is assumed to vanish. It is inferred from the outcomes in Figs. 6-9 that the torsional fundamental frequencies in accordance with both the NLSG and MNSG models decrease as the nonlocal characteristic parameters  $\lambda$  increases, and thus, exhibits a softening behavior in terms of the nonlocal characteristic parameter  $\lambda$  for a given value of  $\mu$ . The gradient parameters  $\mu$  have the effect of increasing the fundamental torsional frequencies that is a larger  $\mu$  involves a larger fundamental frequency for a given value of  $\lambda$ . Therefore, the fundamental frequencies associated with both the NLSG and MNSG models exhibit a stiffening behavior in terms of the gradient parameters  $\mu$ . Remarkably, while the torsional fundamental frequencies of nanobeam associated with the MNSG overestimates the counterpart natural frequencies of NLSG beam with non-vanishing stretch gradient parameter  $\mu_1$ , it is strictly lower than the torsional frequencies of NLSG beam with non-vanishing rotation gradient parameter  $\mu_2$ . This is noticeably occurred as a result of dominant stiffening effects of the rotation gradient parameter  $\mu_2$  compared with the stretch gradient parameter  $\mu_1$  in the framework of NLSG.

As predictable, the torsional fundamental frequency of local elastic beam model is recovered for vanishing nonlocal and gradient parameters.

Numerical values of normalized torsional fundamental frequencies detected in the framework of the nonlocal Lam strain gradient theory and modified nonlocal strain gradient model for doubly clamped and cantilever FG elastic beams are correspondingly reported in Tables 5-6 and 7-8.



Fig. 6. Normalized torsional fundamental frequency of cantilever FG nano-beams by NLSG

 $(\ell_1 \neq 0)$  and MNSG



Fig. 7. Normalized torsional fundamental frequency of doubly clamped FG nano-beams by

NLSG  $(\ell_1 \neq 0)$  and MNSG



Fig. 8. Effects of characteristic parameters on the normalized torsional frequency of cantilever FG nano-beams by NLSG  $(\ell_2 \neq 0)$ 



Fig. 9. Effects of characteristic parameters on the normalized torsional frequency of doubly clamped FG nano-beams by NLSG ( $\ell_2 \neq 0$ )

## 5. Conclusions

As a result of unifying Eringen's nonlocal integral theory and modified strain gradient theory, the nonlocal Lam strain gradient model has been formulated in the present study to examine elasto-static and –dynamic torsional behaviors of FG elastic nano-beams. Three characteristic parameters including two gradient parameters associated with the nano-structure stretch and rotation gradients as well as one nonlocal characteristic parameter, reflecting long-range nonlocal interactions, have been introduced in the nonlocal Lam strain gradient theory.

A variationally consistent constitutive formulation has been conceived, proving thus wellposed torsional problems of FG elastic nano-beams. The nonlocal Lam strain gradient model has been shown to be governed by integral convolutions, conveniently revertible to differential equations equipped with non-classical constitutive boundary conditions. It has been demonstrated that for vanishing nonlocal parameter, the gradient law and non-standard boundary conditions of the modified strain gradient theory are recovered.

Torsional elastostatic rotations and natural frequencies of FG elastic nano-beams have been evaluated utilizing an efficient analytical approach and new numerical benchmarks have been detected. Both softening and stiffening torsional responses can be effectively described in the innovative framework of the novel nonlocal Lam strain gradient model which provides thus an effective approach for design and optimization of structural elements of modern NEMS.

The presented nonlocal Lam strain gradient methodology has been tested by comparing the elasto-static and –dynamic torsional responses of FG elastic nano-beams with the results obtained by the modified nonlocal strain gradient theory.

The main outcomes can be enumerated as follows.

- Elastostatic rotations and torsional natural frequencies of FG elastic nano-beams associated with the nonlocal Lam strain gradient elasticity theory expose softening and stiffening torsional behaviors in terms of the nonlocal and gradient parameters, respectively.
- The torsional response of FG elastic nano-beams consistent with the nonlocal Lam strain gradient model is coincident with the counterpart results associated with the modified nonlocal strain gradient model as the gradient characteristic parameters tend to zero.
- The elastostatic torsional response of FG elastic nano-beams associated with the modified nonlocal strain gradient model underestimates the counterpart results consistent with the nonlocal Lam strain gradient theory with non-vanishing stretch gradient parameter.
   Torsional elastic rotations of nonlocal Lam strain gradient beams with non-vanishing rotation gradient parameter are instead overestimated.
- The torsional fundamental frequency of FG elastic nano-beam associated with the modified nonlocal strain gradient model is strictly higher than the counterpart natural frequency obtained by nonlocal Lam strain gradient theory with non-vanishing stretch

gradient parameter. It is strictly lower than the torsional frequency of nonlocal Lam strain gradient beam with non-vanishing rotation gradient parameter.

- In nonlocal Lam strain gradient elasticity, the stiffening effect of the rotation gradient parameter is more noticeable in comparison with the stretch gradient parameter.
- Torsional elastostatic rotations and fundamental frequencies of local FG elastic beam are recovered as the nonlocal and gradient characteristic parameters tend to zero.

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Table 1. Normalized maximum torsional rotation of doubly clamped FG nano-beam: NLSG

 $(\ell_1 \neq 0)$  vs. MSNG

-										
					$rac{\overline{arphi}_{\max}}{\overline{arphi}_{\max}^{ m LOC}}$			C	6	
		N	$LSG(\ell_1 \neq 0)$	D)				MNSG		
			× ×	,						
λ	$\mu_1 = 0.1$	$\mu_1 = 0.3$	$\mu_1 = 0.5$	$\mu_1 = 0.7$	$\mu_1 = 1.0$	$\mu_{\rm s} = 0.1$	$\mu_{\rm s} = 0.3$	$\mu_{\rm s} = 0.5$	$\mu_{\rm s} = 0.7$	$\mu_{s} = 1.0$
	• 1	• 1	• 1	• 1	• 1	- 3				- 3
$0^+$	0.95743	0.69358	0.44286	0.28687	0.16403	0.92108	0.54261	0.29611	0.17604	0.094552
0.2	2.07504	1.58731	1.02786	0.668904	0.38347	2.02599	1.2541	0.690288	0.411456	0.221314
0.4	3.8313	2.99176	1.94714	1.26924	0.728309	3.76227	2.37216	1.30971	0.781398	0.420513
0.6	6.2262	4.90691	3.20071	2.08789	1.19854	6.12993	3.89679	2.15438	1.28586	0.692147
0.8	9.25973	7.33278	4.78856	3.12483	1.79417	9.12896	5.82798	3.2243	1.92486	1.03622
1.0	12.9319	10.2693	6.7107	4.38009	2.5152	12.7594	8.16574	4.51946	2.69837	1.45272

Table 2. Normalized maximum torsional rotation of doubly clamped FG nano-beam: NLSG

 $(\ell_2 \neq 0)$ 

			$rac{\overline{arphi}_{ ext{max}}}{\overline{arphi}_{ ext{max}}^{ ext{LOC}}}$	-		6
			NLSG (	$\ell_2 \neq 0$ )		
λ	$\mu_2 = 0.01$	$\mu_2 = 0.1$	$\mu_2 = 0.3$	$\mu_2 = 0.5$	$\mu_2 = 0.7$	$\mu_2 = 1.0$
$0^+$	0.94322	0.14245	0.018122	0.0066006	0.0033786	0.0016584
0.2	1.99982	0.302449	0.038486	0.014018	0.007175	0.003522
0.4	3.6602	0.553878	0.070486	0.025673	0.013141	0.00645
0.6	5.92435	0.896735	0.114122	0.041568	0.021277	0.010444
0.8	8.79227	1.33102	0.169395	0.0617	0.031582	0.015502
1.0	12.264	1.85673	0.236304	0.086071	0.044057	0.021625

**Table 3.** Normalized mid-span torsional rotation of cantilever FG nano-beam:  $NLSG(\ell_1 \neq 0)$ 

# vs. MSNG

					$\frac{\overline{\varphi}_{x=1/}}{\overline{\varphi}_{x=1/}^{\text{LOC}}}$	2 2		¢.	6	
		N	$LSG(\ell_1 \neq 0)$	D)				MNSG		
				-						
λ	$\mu_1 = 0.1$	$\mu_1 = 0.3$	$\mu_1 = 0.5$	$\mu_1 = 0.7$	$\mu_1 = 1.0$	$\mu_{s} = 0.1$	$\mu_{s} = 0.3$	$\mu_{s} = 0.5$	$\mu_{s} = 0.7$	$\mu_{s} = 1.0$
$0^+$	0.98581	0.89786	0.81429	0.76229	0.72134	0.97370	0.84754	0.76537	0.72535	0.69819
0.2	1.62501	1.46244	1.27595	1.1563	1.06116	1.60866	1.35137	1.16343	1.07049	1.0071
0.4	2.4771	2.19725	1.84905	1.62308	1.44277	2.45409	1.99072	1.63657	1.46047	1.34017
0.6	3.54207	3.1023	2.53357	2.16263	1.86618	3.50998	2.7656	2.18479	1.89529	1.69738
0.8	4.81991	4.17759	3.32952	2.77494	2.33139	4.77632	3.67599	2.8081	2.37495	2.07874
1.0	6.31064	5.42312	4.2369	3.46003	2.8384	6.25312	4.72191	3.50649	2.89946	2.48424

Table 4. Normalized mid-span torsional rotation of cantilever FG nano-beam: NLSG

 $(\ell_2 \neq 0)$ 

			$\frac{\overline{\varphi}_{x=1/2}}{\overline{\varphi}_{x=1/2}^{\text{LOC}}}$	<u>-</u> 2		¢.,
			NLSG (	$\ell_2 \neq 0$		
-						
λ	$\mu_2 = 0.01$	$\mu_2 = 0.1$	$\mu_2 = 0.3$	$\mu_2 = 0.5$	$\mu_2 = 0.7$	$\mu_2 = 1.0$
$0^+$	0.94334	0.14272	0.018162	0.0066152	0.0033861	0.0016621
0.2	1.54711	0.23415	0.029798	0.010854	0.005556	0.002727
0.4	2.35214	0.356054	0.045314	0.016505	0.008448	0.004147
0.6	3.35843	0.508435	0.064707	0.023569	0.012064	0.005922
0.8	4.56598	0.691293	0.08798	0.032046	0.016403	0.008051
1.0	5.97478	0.904626	0.115132	0.041935	0.021465	0.010536

**Table 5.** Normalized torsional fundamental frequencies of doubly clamped FG nano-beam:

NLSG $(\ell_1 \neq 0)$  vs. MSNG

	$\frac{\overline{\varpi}}{\overline{\varpi}_{\text{LOC}}}$											
	$NLSG(\ell_1 \neq 0)$ MNSG											
λ	$\mu_1 = 0.1$	$\mu_1 = 0.3$	$\mu_1 = 0.5$	$\mu_{\rm l} = 0.7$	$\mu_1 = 1.0$	$\mu_{s} = 0.1$	$\mu_s = 0.3$	$\mu_s = 0.5$	$\mu_s = 0.7$	$\mu_{s} = 1.0$		
$0^+$	1.02393	1.21155	1.51878	1.88811	2.49775	1.04609	1.37139	1.85837	2.41097	3.29032		
0.2	0.686622	0.817239	1.02552	1.27531	1.68738	0.702992	0.92573	1.2552	1.62873	2.22298		
0.4	0.496445	0.591996	0.743019	0.924042	1.22265	0.508701	0.670681	0.909472	1.18015	1.61076		
0.6	0.385248	0.459185	0.576228	0.716575	0.948107	0.394737	0.520155	0.705278	0.915153	1.24905		
0.8	0.313716	0.373568	0.468695	0.582814	0.771099	0.321349	0.42311	0.573627	0.7443	1.01584		
1.0	0.264189	0.314298	0.394267	0.490239	0.648599	0.27053	0.355938	0.482513	0.626059	0.854447		

**Table 6.** Normalized torsional fundamental frequencies of doubly clamped FG nano-beam:

 $\mathrm{NLSG}(\ell_2 \neq 0)$ 

			$rac{ar{\omega}}{ar{\omega}_{_{ m LOC}}}$			
			NLSG	$\ell_2 \neq 0$ )		
λ	$\mu_2 = 0.01$	$\mu_2 = 0.1$	$\mu_2 = 0.3$	$\mu_2 = 0.5$	$\mu_2 = 0.7$	$\mu_2 = 1.0$
$0^+$	1.02762	2.64511	7.41629	12.2886	17.1763	24.5164
0.2	0.681796	1.76013	4.93627	8.1795	11.4329	16.3186
0.4	0.490241	1.26775	3.55592	5.8923	8.23596	11.7556
0.6	0.379948	0.983072	2.75753	4.56935	6.38682	9.11623
0.8	0.309473	0.800805	2.24628	3.72219	5.2027	7.42606
1.0	0.260812	0.67484	1.89292	3.13666	4.38427	6.25788

Table 7. Normalized torsional fundamental frequencies of cantilever FG nano-beam: NLSG

 $(\ell_1 \neq 0)$  vs. MSNG

					$rac{\overline{artail}}{\overline{artail}}_{ m LOC}$	-		C	6	
		N	$LSG(\ell_1 \neq 0)$	0)				MNSG		
λ	$\mu_1 = 0.1$	$\mu_1 = 0.3$	$\mu_1 = 0.5$	$\mu_1 = 0.7$	$\mu_1 = 1.0$	$\mu_s = 0.1$	$\mu_s = 0.3$	$\mu_s = 0.5$	$\mu_s = 0.7$	$\mu_{s} = 1.0$
$0^+$	1.0046	1.0324	1.05739	1.07297	1.08529	1.00881	1.04746	1.07204	1.08408	1.09229
0.2	0.819725	0.858452	0.895339	0.919611	0.939497	0.825677	0.880371	0.91814	0.93752	0.951042
0.4	0.676587	0.724273	0.771502	0.804129	0.831834	0.68401	0.752015	0.802117	0.829042	0.848321
0.6	0.570779	0.621903	0.675341	0.714186	0.748477	0.57862	0.652907	0.711743	0.744967	0.769457
0.8	0.491435	0.542649	0.599054	0.642062	0.681525	0.499123	0.575002	0.639306	0.677421	0.706371
1.0	0.430438	0.480102	0.537358	0.582915	0.626286	0.437734	0.512614	0.579944	0.621704	0.654384

Table 8. Normalized torsional fundamental frequencies of cantilever FG nano-beam: NLSG

 $(\ell_2 \neq 0)$ 

			$rac{ar{\omega}}{ar{\omega}_{_{ m LOC}}}$			
			NLSG	$\ell_2 \neq 0$ )		
λ	$\mu_2 = 0.01$	$\mu_2 = 0.1$	$\mu_2 = 0.3$	$\mu_2 = 0.5$	$\mu_2 = 0.7$	$\mu_2 = 1.0$
0+	1.02856	2.64423	7.41237	12.2819	17.1668	24.5028
0.2	0.833317	2.14541	6.01489	9.9665	13.9305	19.8836
0.4	0.681752	1.75839	4.93068	8.17012	11.4197	16.2998
0.6	0.571601	1.4761	4.13962	6.8594	9.58769	13.6849
0.8	0.490219	1.2669	3.55318	5.88771	8.22953	11.7464
1.0	0.42833	1.10746	3.10614	5.14697	7.19416	10.2686