

Spray parameter comparison between diesel and vegetable oils for non-evaporating conditions in a Constant Volume Combustion Chamber

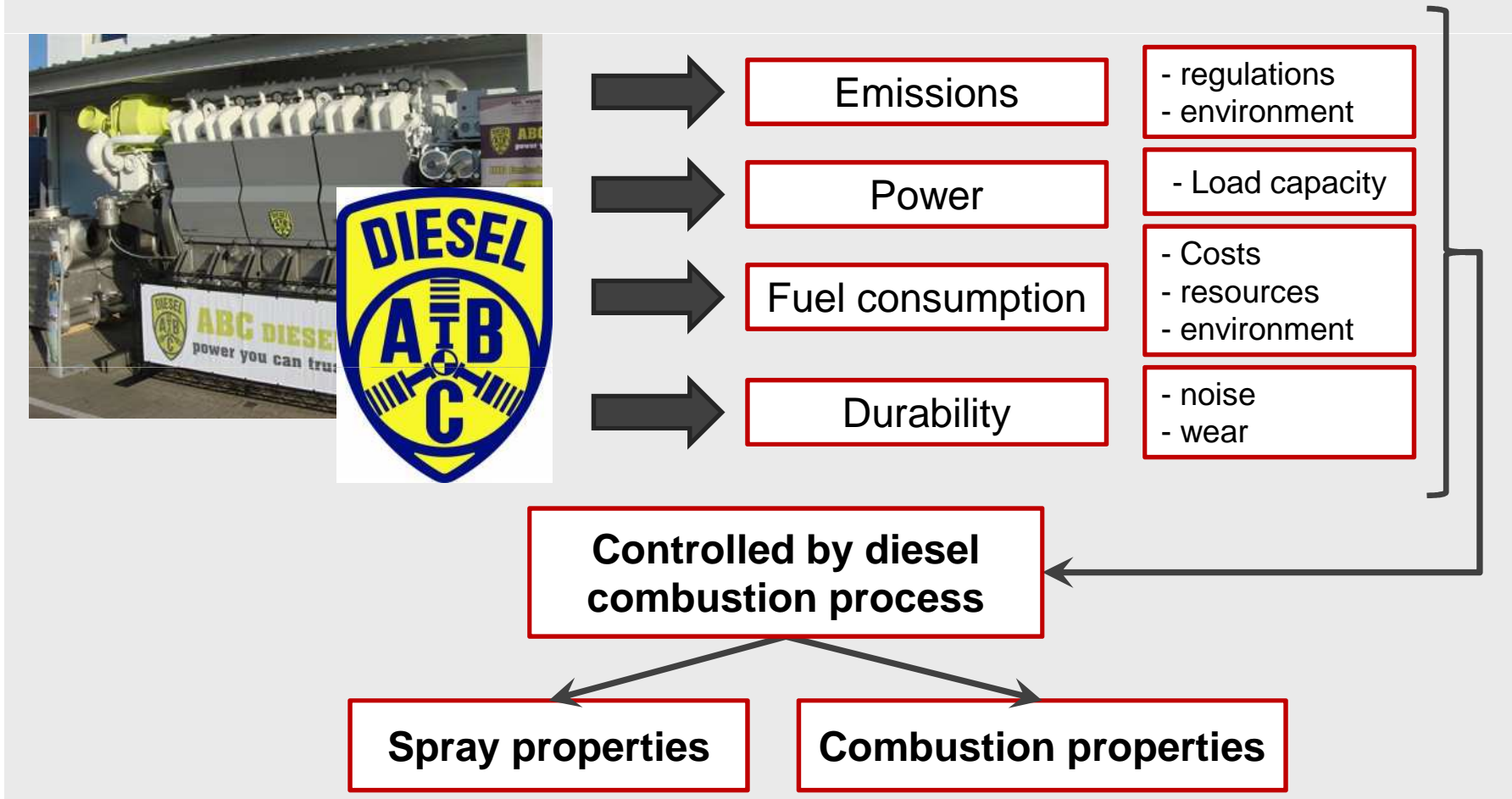
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13-12-2011
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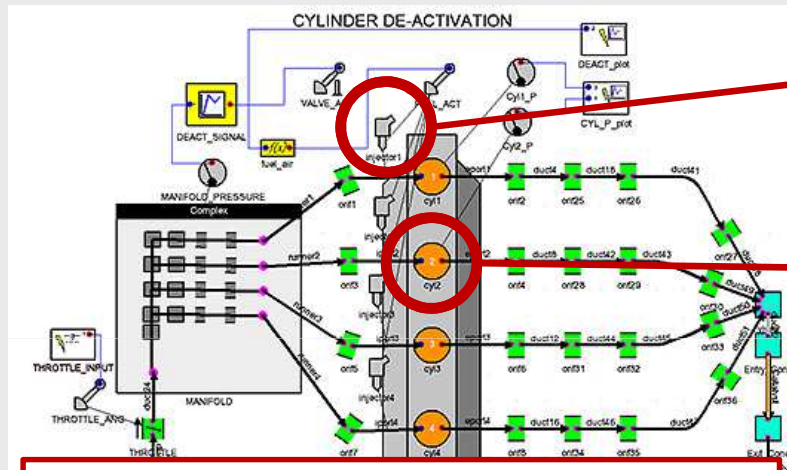
Overview

- Why this research
- Modeling demands
- Why/Which spray parameters
- Why vegetable oils
- Why non-evaporative conditions
- Presented research goal
- Experimental setup
- Remarks to experiments
- Results & discussion
- Conclusions
- Future work

Why this research: project situation



Why this research: goal



Power train modeling

- Emissions
- Power
- Fuel consumption
- Durability

Spray submodel

Combustion submodel

ADAPTATION for

- Variable injection pressure
- Bio-oils (diesel hypothesis valid?)

Engine settings : real-time control
 Engine design : off line design

Why/Which spray parameters

- Why spray parameters

- sprays precede and initiate combustion (premixed & diffused)
- sprays influence the emissions

- Which spray parameters

(depends on the model assumptions)

- Spray angle (related to air entrainment rate & breakup quality)
- Liquid length (related to evaporation speed)
- Penetration length (related to initial combustion behavior)
- Vapor/liquid concentration (related to evaporation speed)
- [Droplet size distribution (related to breakup quality)]

→ Good spray prediction = better prediction of combustion process

Why vegetable oils

- Reduced fuel cost compared to biodiesel
- Lack of knowledge (in engines)
- Implementable on medium speed diesel engines
- Interest from the sector (in Belgium: ABCdiesel, Van Wingen)



Why non-evaporative conditions

- The used hypotheses are strongly linked with the breakup behavior:
 - Mixing limited hypothesis:
 - evaporation dominated by air entrainment
 - spray is saturated @ each moment
 - Droplet size hypothesis:
 - evaporation dominated by droplet size & evaporation
- Current setup limitations

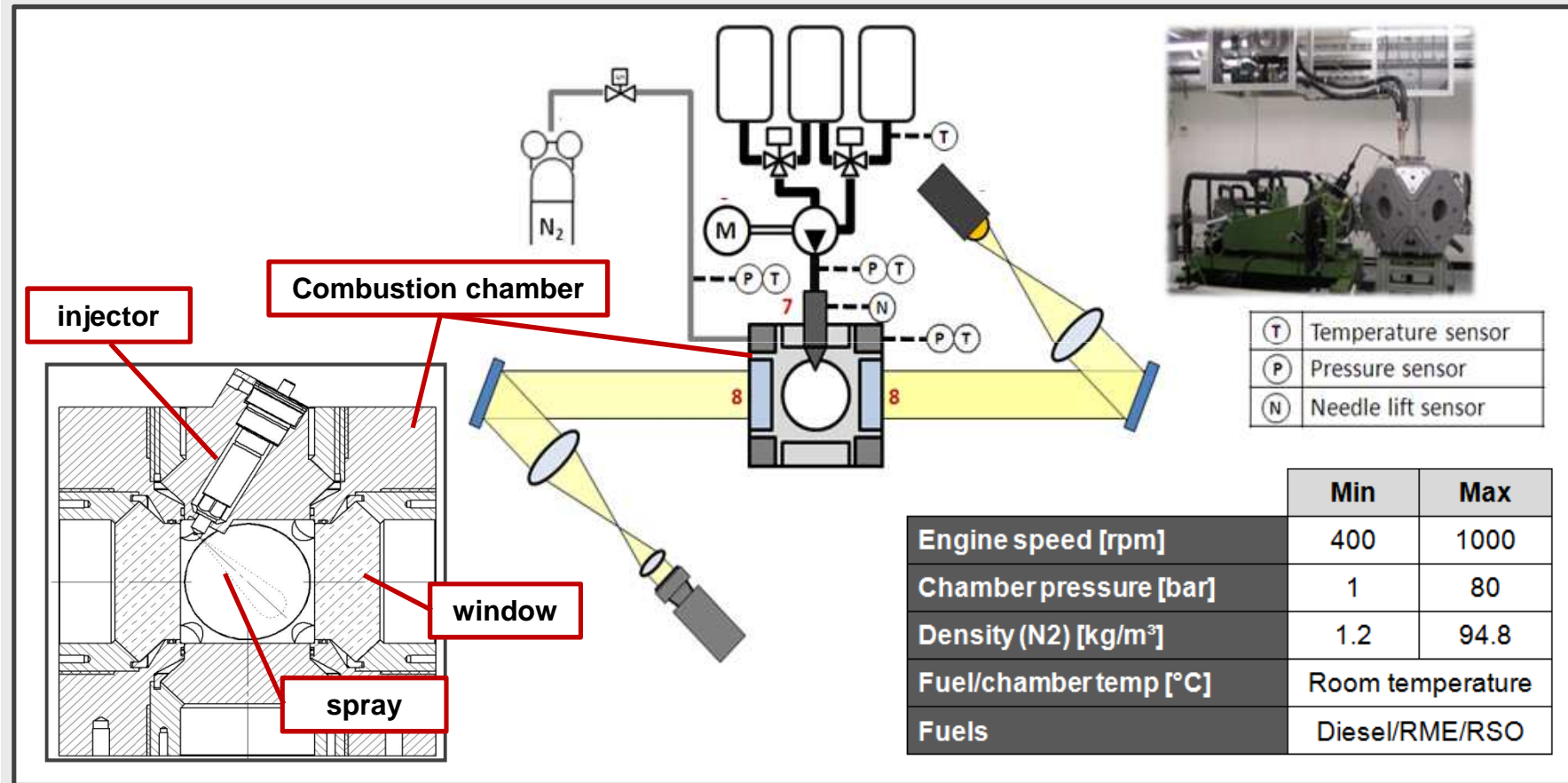
Goal of Presented research

- Qualitative evaluation of physical fuel properties effects on the spray and injection process for 3 fuel types:
 - Conventional diesel
 - Biodiesel (Rapeseed Methylene ester or RME)
 - Straight oil (Rapeseed Oil or RSO)

	Diesel	RME	RSO	Water
Density @15°C [kg/m ³]	835	833	920	998
Kin. viscosity @40°C [mm ² /s]	3.2	3.5-5	33.1	0.658
Surface tension @40°C [mN/m]	27	27.7	>33	69.6
Bulk modulus [GPa]	1.07-1.5	~1.7	1.6-4	2.2

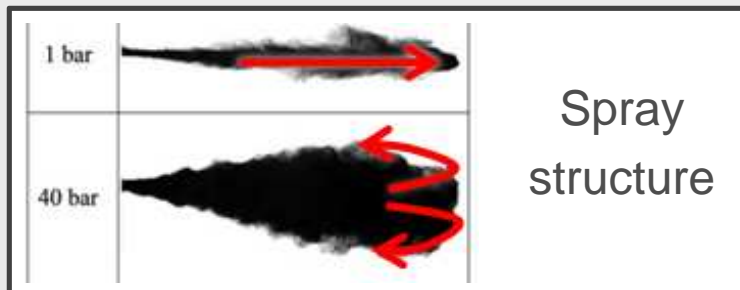
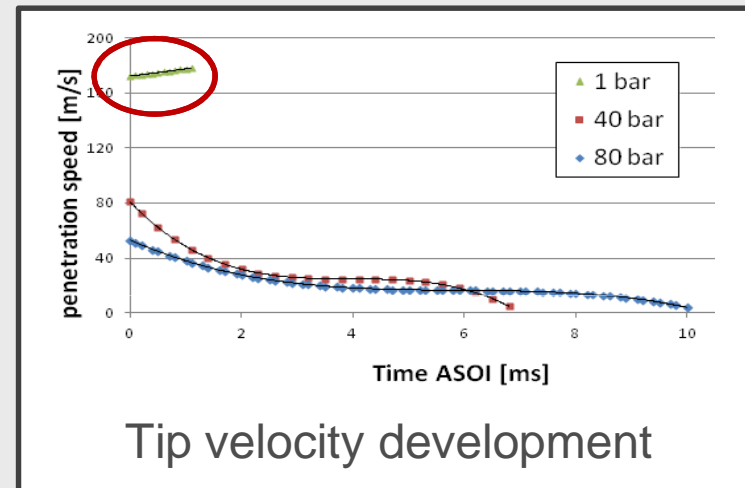
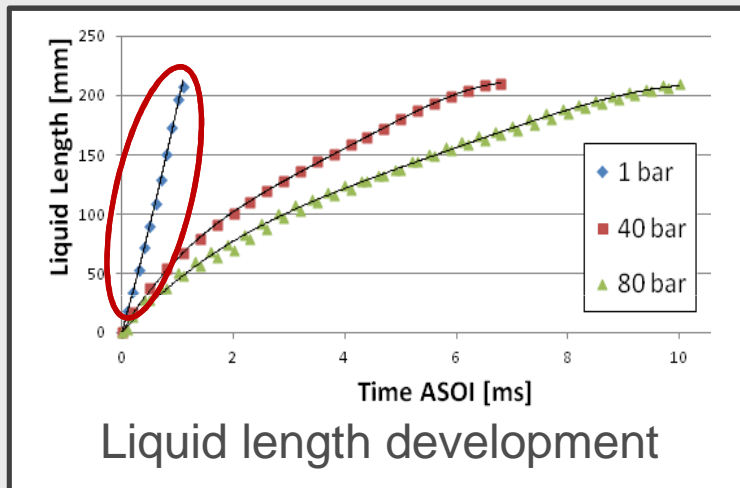
- Can the “Mixing Limited”-hypothesis be applied?

Experimental setup



Remarks about experiments

Air entrainment in sprays depends on chamber density

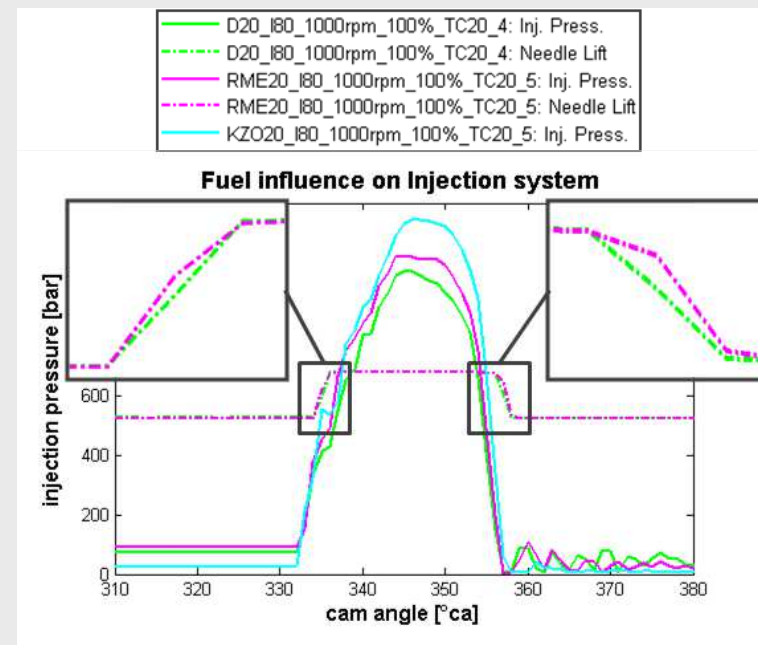
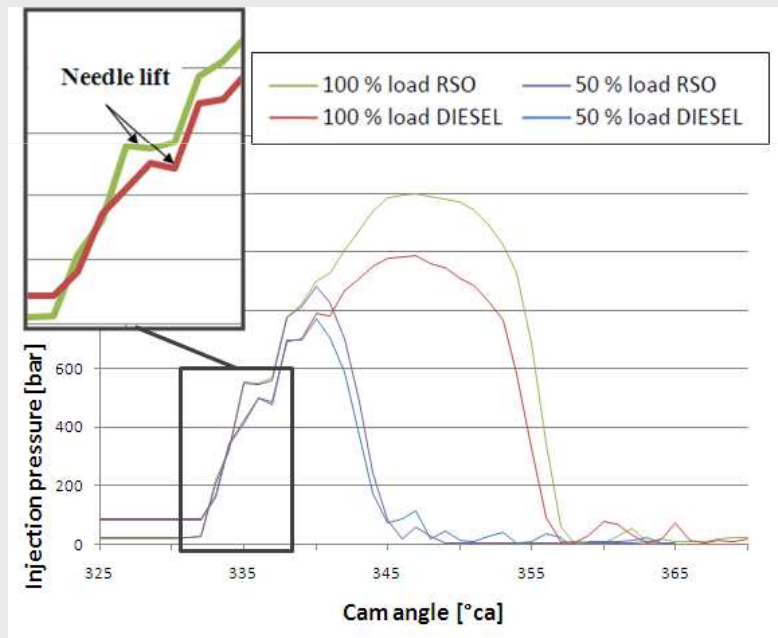


⇒ not all assumptions are valid
⇒ not relevant for engine simulations

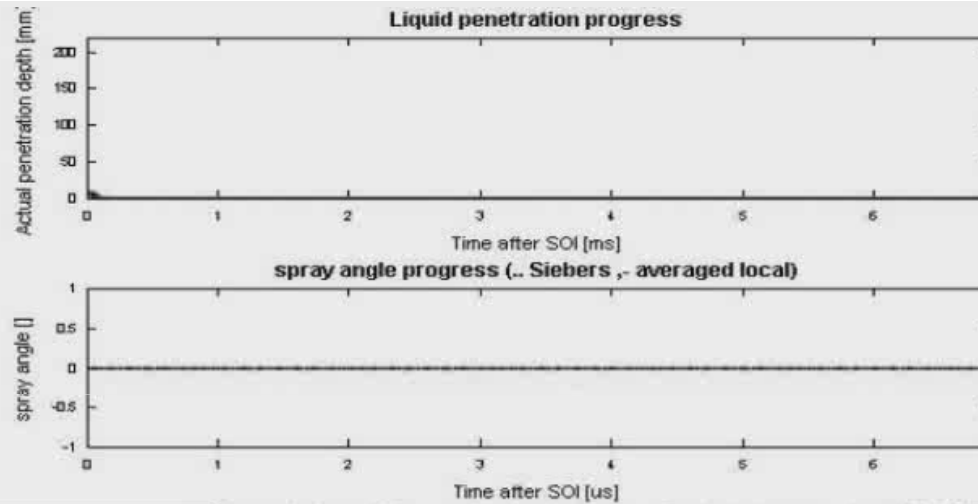
Results & discussion: injection system

- Higher bulk modulus causes
 - Higher pressure build-up
 - Faster needle opening

less important for
Common-rail systems

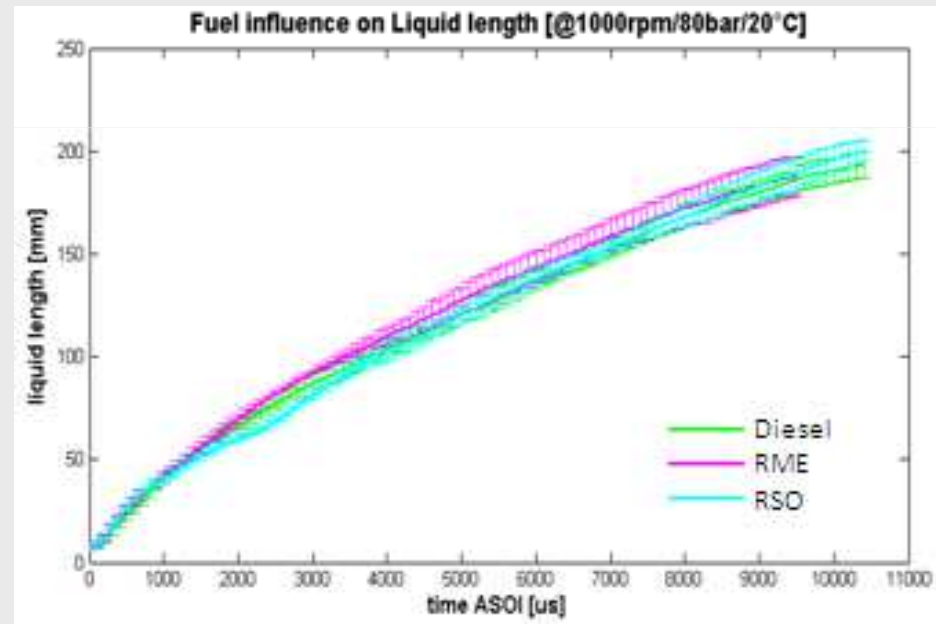
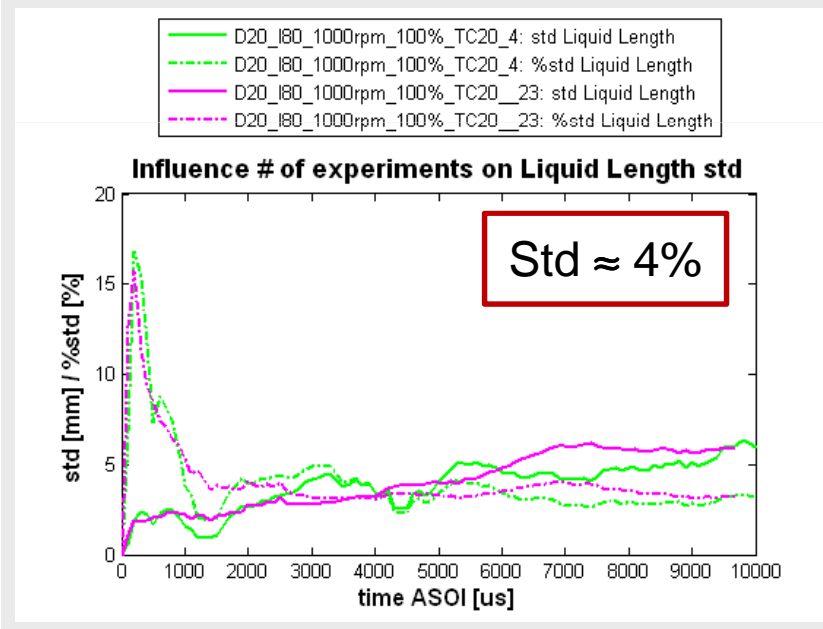


Results & discussion: image processing



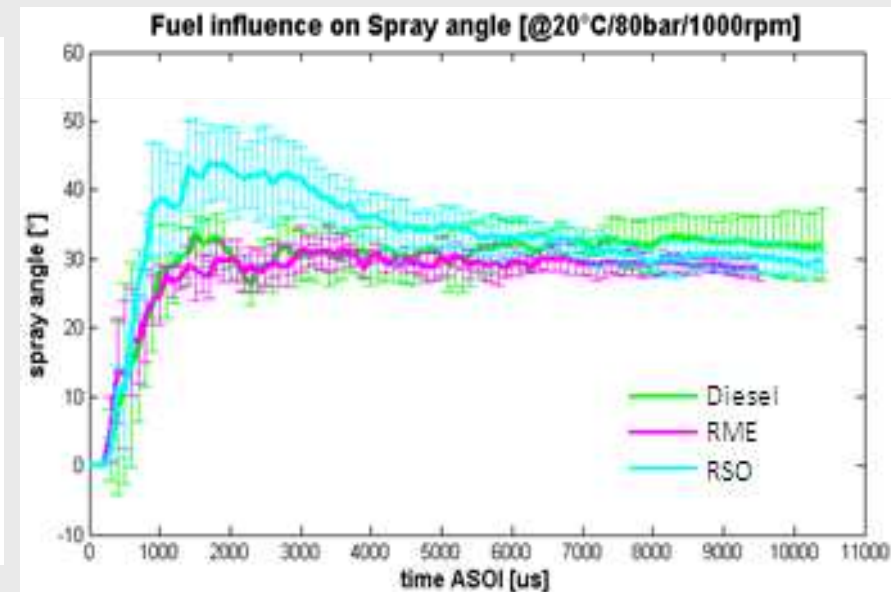
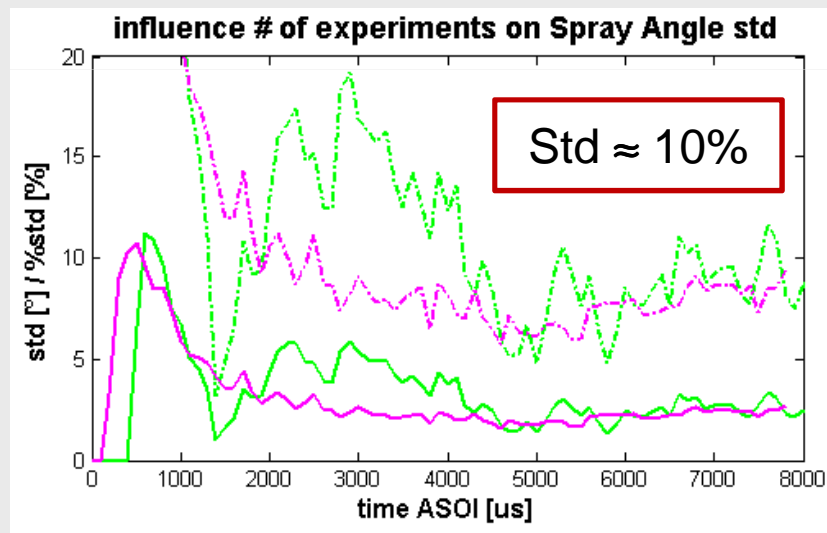
Results & discussion: penetration length

- No significant difference in Liquid penetration for same settings
- Error analysis: standard deviation for more experiments
 - Is more stable
 - Does not decrease



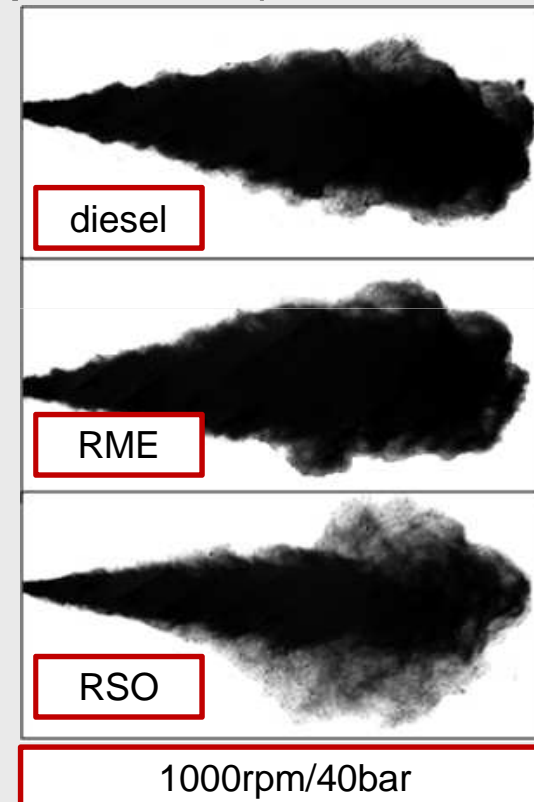
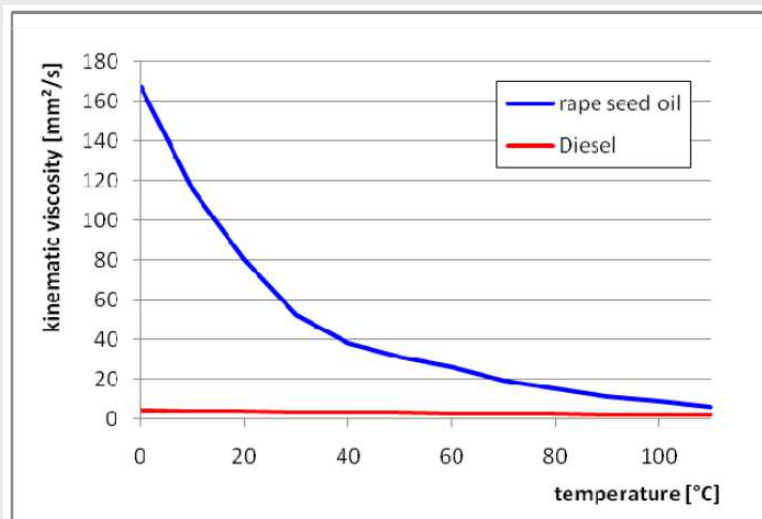
Results & discussion: spray angle

- Initially higher spray angle RSO (higher injection (rate) pressure)
- Error analysis: standard deviation for more experiments
 - Is more stable, but still high
 - Does not decrease



Results & discussion: spray structure

- Similar results for lower rpm (~injection pressure)
- Viscosity & surface tension more important than injection pressure for the break-up quality



Conclusions

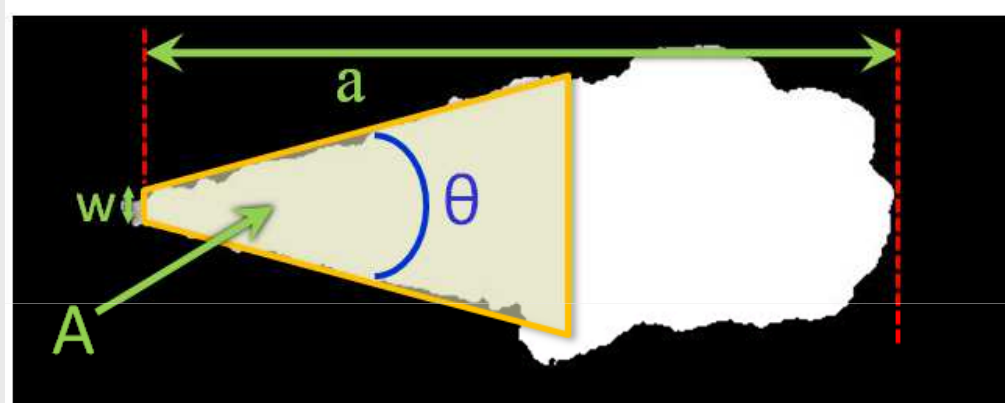
- Difficult to measure spray angle and strongly depends on used definition
- Constant regime spray angle is reasonable, even for
 - variable injection profiles
 - different type of fuels
- “Mixing limited”-hypothesis needs to be questioned
- Large influence of viscosity on ‘Mixing limited’ hypothesis
- additional structure parameter needed for the comparison of spray of different fuels

Future work

- Extension to Evaporative conditions:
 - How the structure affects the evaporation
 - influence of supercritical conditions (case for engines)
- 1D-model adaptation:
 - Variable injection pressure
 - Evaporative conditions: diesel hypothesis valid?
- Higher measuring frequency for sensors/camera
- Higher resolution can give more insight on droplet diameters

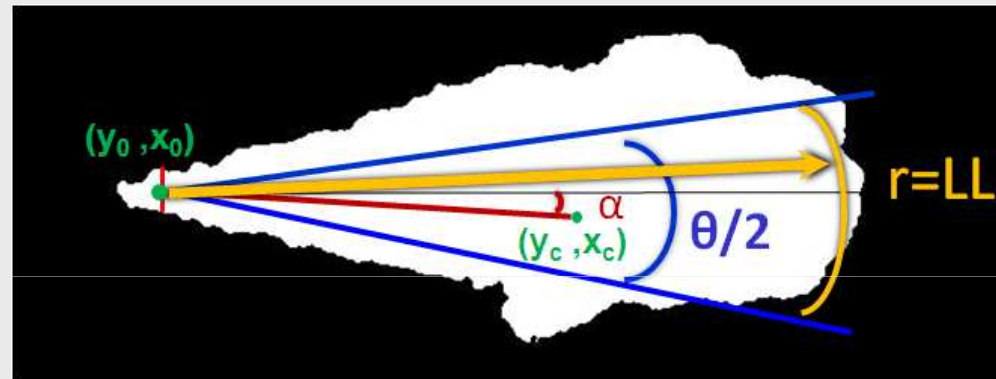
Thanks for your attention

Appendix: spray angle definition



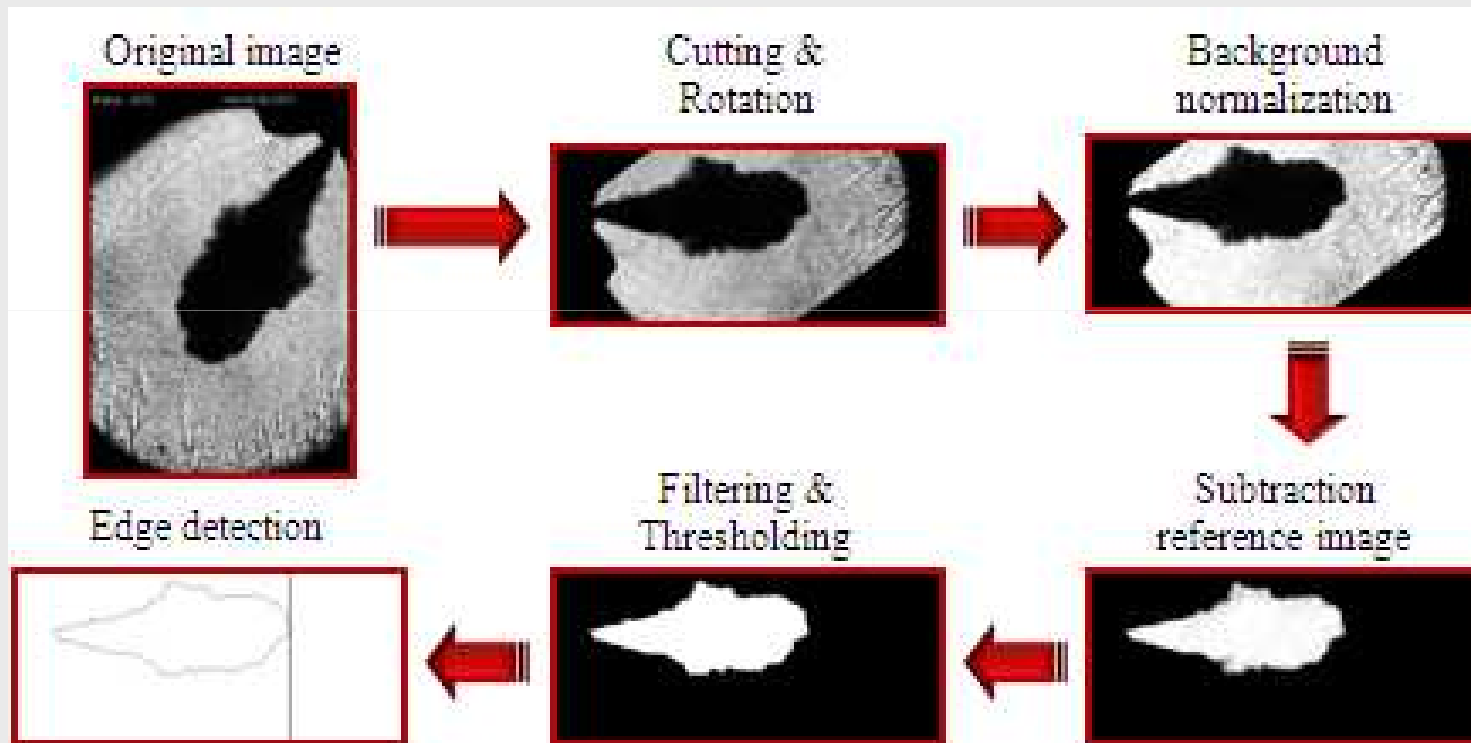
$$\theta = 2 \cdot \tan^{-1} \left(\frac{(A - w \cdot a) / 2}{(a/2)^2} \right)$$

Appendix: liquid length definition



$$\text{real liquid length} = \frac{\text{image liquid length}}{\sin(45^\circ)} + \text{invisible part}$$

Appendix: image processing



Appendix: injector information

Nozzle hole diameter	440 μm
Opening angle holes	150 $^{\circ}$
L/D	2.7
Nozzle shape	cylindrical
Amount of nozzle holes	8
Needle lift	0.65 mm
Sac diameter	2.8 mm
Average mass flow (diesel)*	1.45 mg/cycle
Needle opening pressure	275 bar

*according to engine measurements @1000rpm, full load

Appendix: measurement accuracy

Table 5 – Reproducibility of the measured parameters

Camshaft speed	+/-1,2%
Chamber pressure	+/- 5% (@40bar), +/-2.5% (@80bar)
Chamber temperature	+/-4%
Injection pressure	+/-1% (during the injection)

Table 6 – Accuracy of the image processed data

Start of injection	+100us (@10000fps)
Spray angle std	+/- 10% (20 measurements)
Liquid Length std	+/- 6%
Injection pressure std	+/-2%