



Article Operation Safety of a 2-DoF Planar Mechanism for Arm Rehabilitation

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Abstract: The operation safety of rehabilitation devices must be addressed early in the development process and before being tested on people. In this paper, the operation safety of a 2-DoF (degrees of freedom) planar mechanism for arm rehabilitation is addressed. Then, the safety and efficiency of the device operation is assessed through the Transmission Index (TI) distribution in its workspace. Furthermore, the produced stresses on the human arm are assessed via the FEM (finite element method) when the rehabilitation device reaches five critical positions within its workspace. The TI distribution showed that the proposed design has a proper behaviour from a force transmission point of view, avoiding any singular configuration that might cause a control failure and subsequent risk for the user and supporting the user's motion with a good efficiency throughout its operational workspace. The FEM analysis showed that Nurse operation is safe for the human arm since a negligible maximum stress of $6.55 \times 10^3 \text{ N/m}^2$ is achieved by the human arm when the device is located on the evaluated critical positions.

Keywords: mechanical design; arm rehabilitation; assistive device; neurorehabilitation; operation safety

1. Introduction

Rehabilitation robotics have contributed positively to the rehabilitation processes, speeding up recovery times[1,2] and reducing the physical fatigue of therapists. This has increased the interest in the development of medical devices for motor rehabilitation in the last decades[3–7]. Particularly, several devices have been developed for upper-limb rehabilitation such as exoskeleton and semi-exoskeleton devices[8–14]. Exoskeleton and semi-exoskeleton devices offer large workspaces and customizable exercises. However, they require large links, bulky frames, large motors, and a precise axis alignment with the upper limb joints to avoid injuries. Furthermore, the mechanisms of exoskeletons and semi-exoskeletons must be reconfigured to be used for both left and right arm or else an exoskeleton must be built for each human arm. Regarding portable mechanisms, they generally perform restricted trajectories limited by their mechanical structures [15–17]. Although there are portable end-effector mechanisms that can reproduce customizable trajectories[18,19] they require long links to cover most of the human arm workspace. Some cable-driven devices[20,21] offer convenient advantages regarding portability and customization of the exercises. However, the loss of tension of the cables when the patient makes an oppositional or involuntary movement is an issue that still requires further investigation. Furthermore, cable-driven devices[22] generally need to be over-actuated to maintain the cable tension. On the other hand, the 2-DoF (degrees of freedom) mechanism for upper limb rehabilitation presented in this paper, whose name is Nurse, combines a large number of advantageous features that are hard to find all together in other mechanisms. The mechanism structure of Nurse is composed of light links and a tiny frame



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Copyright: © 2021 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). that facilitates its portability and makes it suitable for home use. Furthermore, the linkage structure offers a large workspace without the need for large and long links. Additionally, it can reproduce customizable and varied exercises for both the left and right human arm without requiring mechanical reconfigurations.

In a previous work, the authors characterized four arm exercises[23] that were later used for a numerical characterization[24,25] and an experimental characterization[26] of Nurse. However, the operation safety of the device had not yet been addressed in[23–26]. Operation safety of the rehabilitation devices should be addressed early in the development process before the device is tested on people[27]. Therefore, the safety operation is an important issue that will be addressed in this paper.

In this paper, the Nurse device is briefly described. Next, the safety and efficiency of the NURSE operation is assessed through the Transmission Index (TI) distribution in the Nurse workspace. After that, the produced stresses on the human arm are assessed via a FEM (finite element method) when Nurse reaches five critical positions within its workspace.

2. Materials and Methods

2.1. A Brief Description of Nurse

Figure 1a shows a scheme of the Nurse device and Figure 1b shows its workspace. The mechanism is composed by mobile links L_1 , L_2 , L_3 , L_4 , L_6 , L_7 , L_8 , L_9 and fixed link L_5 . The point *E*, in Figure 1a, is a tracing point that is follow by the tracing point *F* with an amplification scale of 4. The mechanism has two active joints actuated by motors M_1 and M_2 .



Figure 1. Nurse device: (a) scheme of the linkage structure; (b) mechanism workspace.

Nurse was numerically characterized in [25] and a lab prototype was experimentally characterized in [26]. In [25], the authors showed, through a dynamic analysis and an FEM analysis (finite element analysis), that Nurse is a feasible device for arm rehabilitation. In [26], the authors showed, through an experimental characterization, that Nurse is capable of reproducing different rehabilitation exercises. Figure 2 shows a lab prototype of Nurse and Figure 3 illustrates the user–mechanism interaction. The mechanism structure of Nurse weighs 2.6 kg and fits into a box of $35 \times 45 \times 30$ cm when it is in a contracted position [25,26]. The planar mechanism linkage of Nurse is supported by two wheels that are placed under the tracing point *E* and the amplified tracing point *F*. The trajectory of the user mechanical interface is defined by the amplified tracing point *F*. Nurse is capable of reproducing

different trajectories for upper limb rehabilitation on a plane, including the trajectories for arm rehabilitation designed by the authors in [23], see Figure 4. Figure 4 shows four arm rehabilitation trajectories that were used for the experimental characterization of Nurse in [26]. In Figure 4, Trajectory 1 exercises the shoulder joint, Trajectory 2 the exercises the elbow joint, and Trajectory 3 and 4 exercise the elbow and shoulder joint at the same time. Although Nurse has several safety features, such as digital and physical emergency buttons, over current protections and the possibility of remembering the trajectories customized by the therapist for a particular patient, further safety analysis should be performed before experimenting with humans. Furthermore, a characterization of the workspace based on a transmission index is required for a better device performance.



Figure 2. A lab prototype of Nurse device.



Figure 3. User-mechanism interaction.



Figure 4. Trajectories for upper limb rehabilitation: (a) Trajectory 1; (b) Trajectory 2; (c) Trajectory 3; (d) Trajectory 4.

2.2. Performance Analysis

To ensure safe and efficient use of the Nurse device, the torque of the actuators needs to be transmitted to the operator's arm with a smooth and controlled behavior. While a safe behavior is intrinsic to mechanisms with compliant elements or driven by flexible tendons [28], rigid-link mechanisms such as Nurse require a careful motion planning and force transmission analysis [29]. Since the pantograph mechanism only amplifies the behavior of the actuated 5-bar linkage that generates the motion, the critical components for force transmission are links L_1 and L_4 , as the two links that are directly actuated by the rotational motors, and L_1 and L_4 , which transmit motion from the motors to path generating point E, as per Figure 1.

From a kinematic point of view, Nurse is characterized by two degrees of freedom actuated by two coaxial rotational motors (M_1 and M_2 in Figure 1), which directly define the motion of point E that is then amplified in point F by the pantograph. Thus, the architecture of the 5-bar linkage is characteristic of planar parallel mechanisms, with two independently actuated limbs (first limb: L_1 to L_2 , second limb: L_4 to L_3) that converge in point *E*. As reported in [25], the motion of point *E* can be described as:

$$x_E = H(\cos\theta_1 + \cos\theta_2) \tag{1}$$

$$y_E = H(\sin\theta_1 + \sin\theta_2) \tag{2}$$

where the geometrical parameter $H = L_1 = L_2 = L_3 = L_4 = L_5$ is fixed, and the angles θ_1 and θ_2 represent the actuation of M_1 and M_2 , respectively. The position of point E can be defined by an angle α , as illustrated in Figure 5, which can be evaluated as:

$$\alpha = \tan^{-1} \frac{(y_E - H)}{x_E} = \tan^{-1} \frac{(y_F - H)}{x_F}$$
(3)

and a distance r_2 , which can be computed from Equations (1) and (2) as:

$$r_2 = \sqrt{(y_E - H)^2 + x_E^2} \tag{4}$$



Figure 5. Kinematic diagram of Nurse with motion and design parameters [25].

The forward kinematics of the full mechanism (up to point *F*) can be then obtained from the distance r_1 from the amplification factor of the pantograph *k* as:

$$r_1 = kr_2 = \frac{L_9}{L_8} r_2 \tag{5}$$

The position of point *F* is thus given as:

$$x_F = r_1 \cos \alpha = \frac{L_9 \sqrt{(y_E - H)^2 + x_E^2}}{L_8} \cos\left(\tan^{-1}\frac{(y_E - H)}{x_E}\right)$$
(6)

$$y_F = H + r_1 \sin \alpha = \frac{L_9 \sqrt{(y_E - H)^2 + x_E^2}}{L_8} \sin\left(tan^{-1}\frac{(y_E - H)}{x_E}\right)$$
(7)

This structure results in a non-linear relationship between the torques in the actuators and the force transmitted to point E (and, through that, to the extremity of the pantograph F). For this reason, an analysis of the performance of the 5-bar linkage is here proposed to

characterize the transmission behavior of Nurse. The Transmission Index (TI) quantifies the relationship between the transmission wrench screw and the output twist screw, as explained by Chen and Angeles in [30]. The TI is widely recognized as one of the best indices to evaluate the static performance of parallel mechanisms [31] and to analyze their behavior in their reachable workspace [32], as singularities can be identified from discontinuities and null values for the TI. Some examples of applying the TI to parallel architectures are reported in [33–35]. In particular, the procedure developed in [34,35] is adopted in this research to evaluate the TI, with reference to the kinematic scheme of the motion-generating 5-bar linkage of Nurse in Figure 6. This procedure requires the determination of a pressure angle α_i , which is defined as the angle the direction of the force applied to a link and the resulting motion and can be evaluated for each degree of freedom of the mechanism with the following steps:

- 1. Locking the actuators of all the limbs of the parallel mechanism except for the one corresponding to the degree of freedom under analysis (limb *i*).
- 2. Substituting the non-locked actuator and its limb with the corresponding unit force transmitted to the end-effector.
- 3. Evaluating the instantaneous velocity corresponding to the motion of the end-effector resulting from the unit force applied in the previous point.



Figure 6. Kinematic diagram of the 5-bar linkage of Nurse with main twist and wrench directions: (**a**) pressure angle of the first limb in an example configuration; (**b**) pressure angle of the second limb in an example configuration.

The pressure angle α_i , relative to limb *i* and obtained as the angle between the force of point 2 and the velocity of point 3, can be used to obtain the force transmission efficiency η_i in that limb as:

$$\eta_i = \cos \alpha_i \tag{8}$$

After evaluating η_i for each limb, the output transmission index of the mechanism can be determined as:

$$TI = \min(\eta_i) \ \forall i \tag{9}$$

In Nurse's 5-bar linkage, the links L_2 and L_3 can only transmit forces along the link's direction under the assumption of static balance with negligible friction and inertial effects. Thus, the direction of the unit force that these links can transmit to the end-effector is

defined by the orientation of the link itself, which is θ_1 for link L₂ (always parallel to input link L₄) and θ_2 for link L₃ (always parallel to input link L₁).

When one of the parallel limbs of the 5-bar linkage is removed and the remaining actuator motion fixed, the only movable limb is L_2 if removing the limb defined by L_4 and L_3 , or L_3 if removing the limb defined by L_1 and L_2 . These movable limbs can only rotate around the revolute joints that connect them to their respective input links, resulting in an instantaneous velocity that is normal to the orientation of the link. Thus, the instantaneous velocity obtained when removing the limb defined by L_4 and L_3 is defined by an orientation of $\theta_1 + \pi/2$, whereas the instantaneous velocity obtained when removing the limb defined by L_4 and L_3 is defined by an orientation of $\theta_2 + \pi/2$.

Thus, the pressure angles corresponding to these two cases are:

$$\alpha_1 = \theta_2 - \theta_1 + \frac{\pi}{2}; \ \alpha_2 = \theta_1 - \theta_2 + \frac{\pi}{2};$$
(10)

From Equation (10), the pressure angles are always equal in modulus (even if their sign can vary, depending on how they were defined). This means that the efficiency of the two limbs, defined as $\eta_i = \cos \alpha_i$ as per Equation (8), is always the same due to the symmetry of the system. Therefore, the TI of the system in a given configuration can be computed as:

$$TI = \cos\left(\theta_2 - \theta_1 + \frac{\pi}{2}\right) \tag{11}$$

2.3. Operation Safety

A linear static FEM (finite element method) analysis has been performed in order to assess the produced stress on the human arm when Nurse device reaches critical positions within its workspace. Five critical positions within a mechanism workspace can be chosen by tracing an imaginary rectangle covering most of the workspace (Norm ISO 9283 [36]). The central point and the four corners of the imaginary rectangle are five critical positions. Figure 7 shows five critical positions within the Nurse workspace, where point *a*, point *b*, point *c*, and point *d* are the corners of the imaginary rectangle and point *e* is the central point of the rectangle. Figure 8 shows a CAD (computer-aided design) of the Nurse device that emulates the real lab prototype. Figure 9a–e show the Nurse device on the positions *a* (-496, 700), *b* (496, 700), *c* (-496, 148), *d* (496, 148), *e* (0, 424), respectively.



Figure 7. Five critical positions (*a* (-496, 700), *b* (496, 700), *c* (-496, 148), *d* (496, 148), *e* (0, 424)) within the NURSE workspace.



Figure 8. A CAD (computer-aided design) of Nurse device.



Figure 9. Nurse device on critical positions: (a) critical position a (-496, 700); (b) critical position b (496, 700); (c) critical position c (-496, 148); (d) critical position d (496, 148); (e) critical position e (0, 424).

During the simulation, a 1060 aluminum alloy has been used for Nurse bars and wheels and ABS (acrylonitrile butadiene styrene) has been used for the user mechanical interface. Table 1 shows the main material characteristics of the 1060 aluminum alloy and Table 2 shows the main characteristics of the ABS. It is important to note that the materials used during simulations correspond to the materials used in practice for the lab prototype of Nurse in Figures 2 and 3.

Properties	Units	1060 Aluminum
Elastic Modulus	N/m ²	$6.90 imes 10^{10}$
Poisson's Ratio	N/A	0.33
Shear Modulus	N/m^2	$2.70 imes 10^{10}$
Mass Density	kg/m ³	2 700
Tensile Strength	N/m^2	$6.89 imes10^7$
Yield Strength	N/m^2	$2.76 imes 10^7$

Table 1. Main characteristics of 1060 aluminum alloy.

Table 2. Material characteristics of the ABS (Acrylonitrile Butadiene Styrene) material.

Properties	Units	ABS ¹
Elastic Modulus	N/m^2	$0.2 imes 10^{10}$
Poisson's Ratio	N/A	0.39
Shear Modulus	N/m^2	$0.32 imes 10^{10}$
Mass Density	kg/m ³	1020
Tensile Strength	N/m^2	$3 imes 10^7$

¹ ABS (acrylonitrile butadiene styrene).

Hyperelastic materials can be used to simulate the biological skeletal muscle behavior [37]. Particularly, silicone materials have the ability to imitate the distribution of stress in muscle tissue [38]. Therefore, to carry out the FEM analysis, a human arm has been simulated using silicone. Table 3 shows the main material characteristics of the used silicone. The weight of the simulated human arm part (hand + forearm part) has been assumed as 1.18 Kg according to the average weight of a human arm [39,40]. Figure 10 shows the simulated human arm. The used CAD model of the human arm with articulated fingers is an open access model available online in [41].

During the FEM simulation, gravity force acts along the Z axis. In addition, a torque of 3.53 Nm has been used for each motor of Nurse device (M_1 and M_2). The motor torque of 3.53 Nm is the maximum torque required by the motors of Nurse prototype during the reproduction of arm rehabilitation exercises according to the experimental tests published in [26].

Table 4 shows the main characteristics of the mesh used for the FEM analysis. The Von Mises stress function has been used to measure the three main stresses acting on X, Y, and Z axes of the human arm. The Von Mises stress function σ_{vm} is expressed as [42]:

$$\sigma_{Von\,Mises} = \sqrt{\frac{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2}{2}}$$
(12)

where:

 σ_1 = principal stress acting on X. σ_2 = principal stress acting on Y.

 σ_3 = principal stress acting on Z.

Properties	Units	Silicone
Elastic Modulus	N/m ²	$11.24 imes10^{10}$
Poisson's Ratio	N/A	0.28
Shear Modulus	N/m^2	$4.9 imes10^{10}$
Mass Density	kg/m^3	2330
Yield Strength	N/m^2	$12 imes 10^7$

Table 3. Material characteristics of the used silicone material.



Figure 10. Simulated human arm.

Table 4. Mesh information for FEM (finite element method) analysis.

Mesh Type	Solid Mesh
Mesh type	Mixed Mesh
Mesher Used	Curvature-based mesh
Jacobian points	16 Points
Maximum element size	15.40 mm
Minimum element size	3.08 mm
Total Nodes	264,679
Total Elements	157,419

3. Results and Discussion

3.1. Performance Analysis

By using the forward kinematics in Equations (6) and (7), the TI in Equation (11) can be mapped in the workspace of the Nurse device. In order to obtain operational parameter values, the design values defined in [25] are adopted for this evaluation, as summarized in Table 5. By using those values, Equation (11) was computed throughout the reachable workspace of Nurse, as shown by the colormap in Figure 11.

Table 5. Design parameters of the Nurse prototype.

Variable	Value	Units
L_1	180	mm
L_2	180	mm
L_3	180	mm
L_4	180	mm
L_5	180	mm
L_6	360	mm
L_7	270	mm
L_8	90	mm
L_9	360	mm



Figure 11. Transmission Index distribution in the workspace.

The workspace in Figure 11 does not represent the full amplified workspace of the 5-bar linkage illustrated in Figure 6, since its motion is physically constrained by the length of the pantograph's links. Therefore, only a central region of the full workspace of the 5-bar linkage is actually reachable, and this reflects into the workspace in Figure 6. It is possible to observe that the central region of the workspace is characterized by a good performance, with optimal force transmission in those points that corresponds to a perfect square configuration of the 5-bar linkage (i.e., when $\theta_2 - \theta_1 = \pi/2$), with the TI reaching its maximum allowed value of 1. The two regions above and below these optimal values, which all lie in an arc of circumference, are characterized by values that become gradually smaller the more they are distant from the optimal region.

The central region of the workspace is characterized by a TI value of 0.8 and covers more than 75% of the reachable workspace. In the remaining regions, at the top and the bottom of the workspace in Figure 11, the TI slowly decreases to a minimum value of 0.0056. While this represents a point of extremely low efficiency, it is important to note that the TI never reaches a value of zero. Since the TI of a mechanism becomes zero in singular configurations, this means that the physical constraints to the motion of the 5-bar linkage imposed by the pantograph not only have the effect of reducing its motion range, but also prevent it from reaching any singular configuration (a desirable outcome).

Overall, this result demonstrates that the proposed design performs well from a force transmission point of view, avoiding any singular configurations that might cause a control failure and subsequent risk for the user, and supporting the user's motion with a good efficiency throughout its operational workspace.

3.2. Operation Safety

Human arm stresses have been obtained from the FEM analysis. Figure 12 shows the stress values on the human arm when Nurse reaches the critical positions *a* (-496,700), *b* (496,700) and *c* (-496,148). Figure 13 shows the stress values on the human arm when Nurse reaches the critical positions *d* (496,148) and *e* (0,424). Table 6 shows the maximum stress values on the human arm when Nurse reaches the five critical positions.



Figure 12. Stress results when Nurse reaches critical positions: (a) critical position a (-496,700); (b) critical position b (496,700); (c) critical position c (-496,148).



Figure 13. Stress results when Nurse reaches critical positions: (a) critical position d (496, 148); (b) critical position e (0, 424).

Fable 6. Maximum stress values on the hu	iman arm when Nurse	reaches five critical	l positions
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Critical Positions	Maximum Stress Values
a (-496, 700)	$3.99 imes10^3\mathrm{N/m^2}$
b (496, 700)	$4.41 imes 10^3~\mathrm{N/m^2}$
c (-496, 148)	$6.53 imes10^3\mathrm{N/m^2}$
d (496, 148)	$6.55 imes10^3\mathrm{N/m^2}$
e (0, 424)	$4.43 imes10^3\mathrm{N/m^2}$

Skeletal muscles have a complex nature, and some mechanical properties cannot be estimated in a noninvasive way [43]. Therefore, these quantities are estimated using mathematical models [43]. According to the main mechanical models of skeletal muscles [43–46], a muscle has a maximum peak stress of $668.80 \times 10^3 \text{ N/m}^2$ [44–46], a maximal contractile stress of $30 \times 10^3 \text{ N/m}^2$ [47], and a maximum isometric stress of $300 \times 10^3 \text{ N/m}^2$ [48,49].

The FEM analysis results, in Figures 12 and 13, and Table 6, show that the human arm achieves a maximum stress value when the Nurse device reaches the critical position *d* (496,148) with a maximum stress of 6.55×10^3 N/m². Therefore, stress values on the human arm when Nurse reaches critical positions are negligible with respect to the mentioned mechanical properties of the skeletal muscles. On the other hand, the achieved stress values are negligible with respect to the average bending strength (the material's ability to withstand stress) of the humerus (128,430 × 10³ N/m²), ulna (135,160 × 10³ N/m²), and radius (80,310 × 10³ N/m²) [50]. In summary, the stress results show that the Nurse operation is safe for the human arm.

4. Conclusions

In this paper, an assistive device for the upper limb, called Nurse, is presented and characterized with numerical simulations. The proposed mechanism is made of a five-bar linkage that is actuated by two motors and generates a trajectory for a user's arm. The motion of the five-bar linkage is amplified by a pantograph to obtain a wide workspace that covers a typical human arm's mobility while maintaining a good force transmission efficiency, as shown by the transmissibility analysis in this manuscript. Furthermore, operation safety is demonstrated with a finite element analysis in five critical positions for the proposed assistive device: four at the borders of its operational workspace, and one at its center. The results of both the finite element analysis and the Transmission Index evaluation validate the feasibility of using the Nurse design and prove both its efficiency and safety in assisting and guiding exercises for arm rehabilitation.

5. Patents

Chaparro-Rico, B. D. M.; Cafolla, D.; Ceccarelli, M.; Castillo-Castaneda, E. Device for arm motion assistance, Italian Patent No. 102016000107499, granted on 12 March 2019.

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