

May 2022

## FEA analysis on a beam of Hexacopter Drone Frame

aicha chahbi el alaoui  
*Mississippi State University, ac3849@msstate.edu*

taymae ben messaoud  
*Mississippi State University, tb2573@msstate.edu*

Follow this and additional works at: <https://scholarsjunction.msstate.edu/fea>



Part of the [Mechanical Engineering Commons](#)

---

### Recommended Citation

chahbi el alaoui, aicha and ben messaoud, taymae, "FEA analysis on a beam of Hexacopter Drone Frame" (2022). *ME 4233/6233 Fundamentals of FEA*. 36.  
<https://scholarsjunction.msstate.edu/fea/36>

This Infographic is brought to you for free and open access by the College of Engineering, James Worth Bagley at Scholars Junction. It has been accepted for inclusion in ME 4233/6233 Fundamentals of FEA by an authorized administrator of Scholars Junction. For more information, please contact [scholcomm@msstate.libanswers.com](mailto:scholcomm@msstate.libanswers.com).

# FEA Analysis on a Beam of a Hexacopter Drone Frame

---



# Presentation Outline



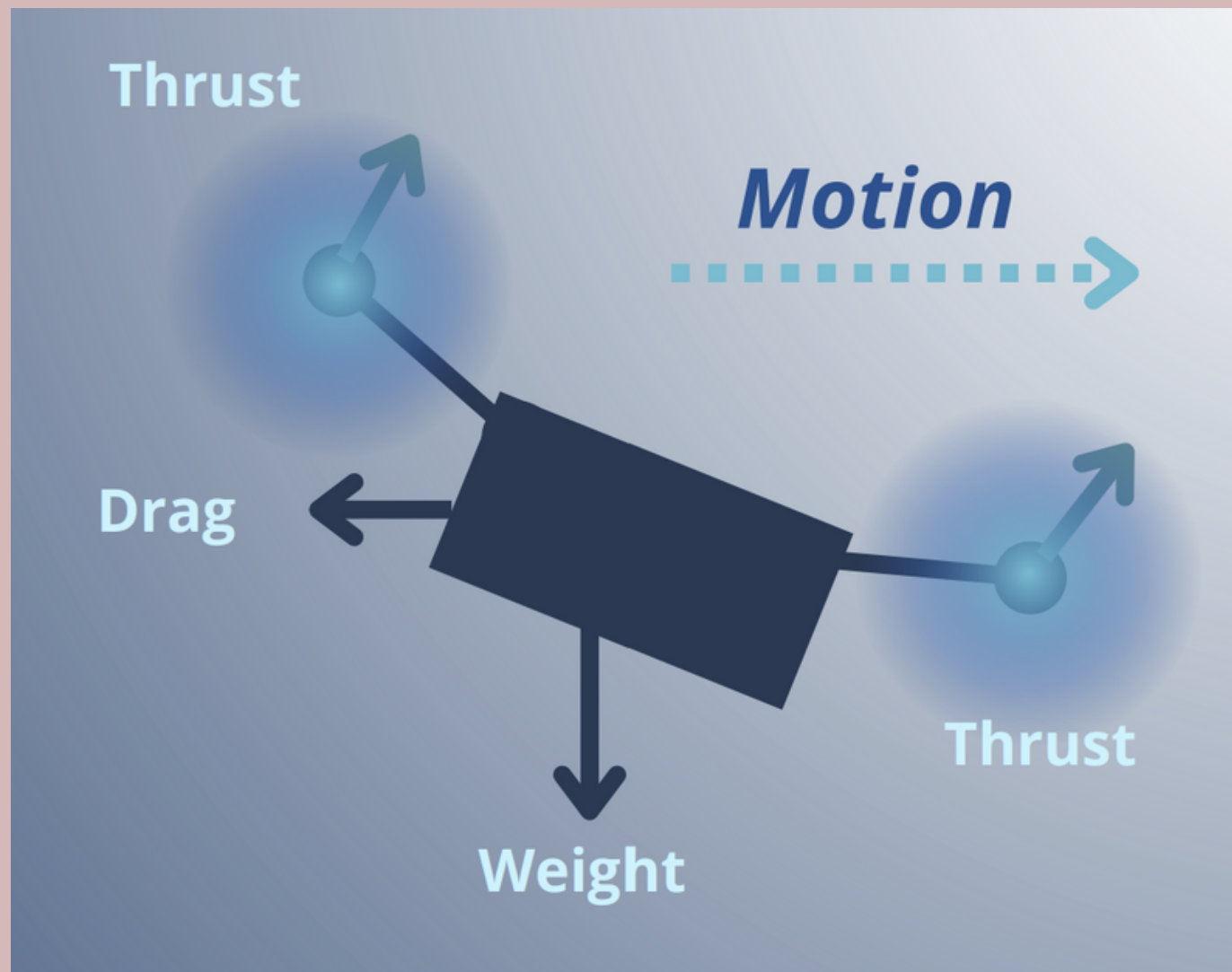
- Scope
- Drone physics
- Material Selection
- Design Reference
- Assumptions
- Calculations
- Simulation
- Results

# SCOPE

*Perform an analysis on the behavior of the beam constituting the hexacopter drone*

- THE MEASUREMENTS OF A STANDARD DRONE
- LOAD ANALYSIS
- FAILURE SIMULATION
- RESULTS

# DRONE PHYSICS



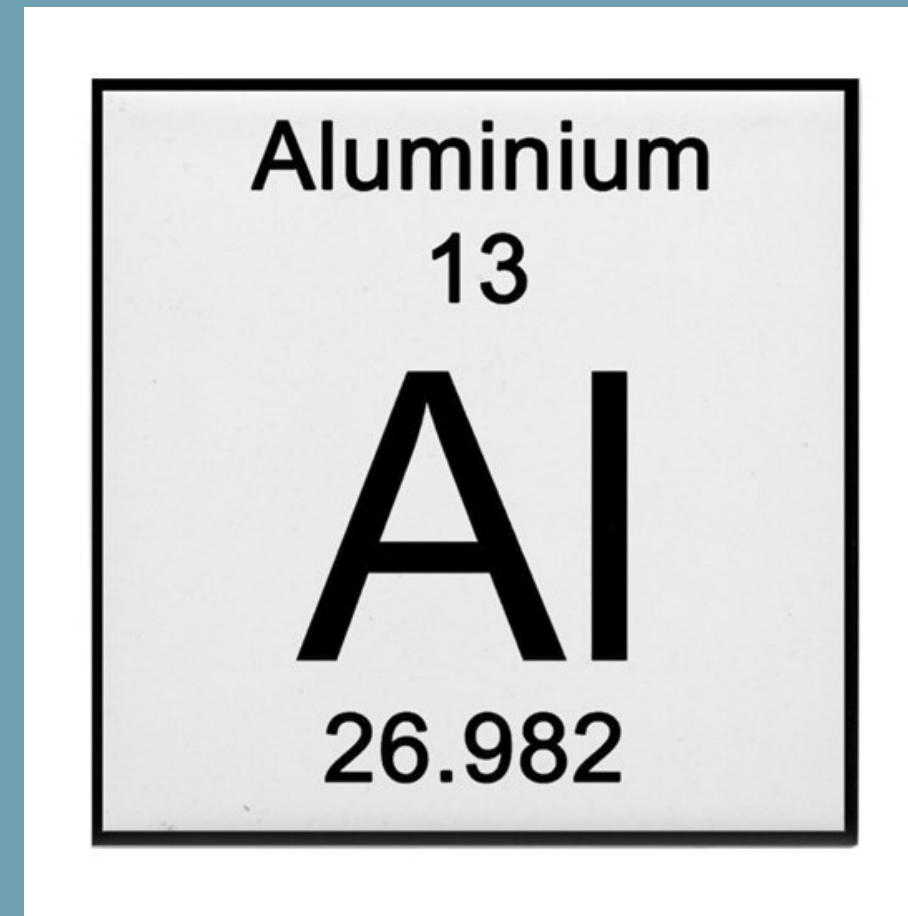
## ***MECHANISM***

- A machine that flies with the help of a remote-control system
- The aerodynamic shape of the drone is selected

## ***PROBLEMATIC***

- To identify the regions of stress concentration and why is it increased in that area in order to reduce it

# MATERIAL SELECTION



|                       |                       |
|-----------------------|-----------------------|
| Failure Stress        | 276 MPA               |
| Failure Strain        | 207 MPA               |
| Factor of Safety      | 2                     |
| Density               | 2,7 g/cm <sup>3</sup> |
| Modulus of Elasticity | 68,9 GPA              |
| Poisson's Ratio       | 0,33                  |

# Design Reference



|                       |                        |
|-----------------------|------------------------|
| Frame Weight          | 810g                   |
| Arms Diameters        | 16mm                   |
| Motor to Motor        | 695mm                  |
| Center Body Dimension | 195x195x2 mm           |
| All up Weight         | 2.8kg                  |
| Payload               | 1.8-2kg                |
| Propellers            | 13in                   |
| Thrust per arm        | 800-1000g              |
| Material              | PCB                    |
| Density               | 1.850g/cm <sup>3</sup> |

# Assumptions

- The twist is neglected, as it is small in a drone
- The angle formed between every two consecutive arms is  $60^\circ$ , for the sake of symmetry;
- The material used is isotropic (homogeneous) ( $E = \text{constant} \rightarrow \varepsilon_{xy} = \varepsilon_{yx}$ ); Friction with air neglected.
- We assume that the deflection is small, because the smaller the deflection, the better the stability.

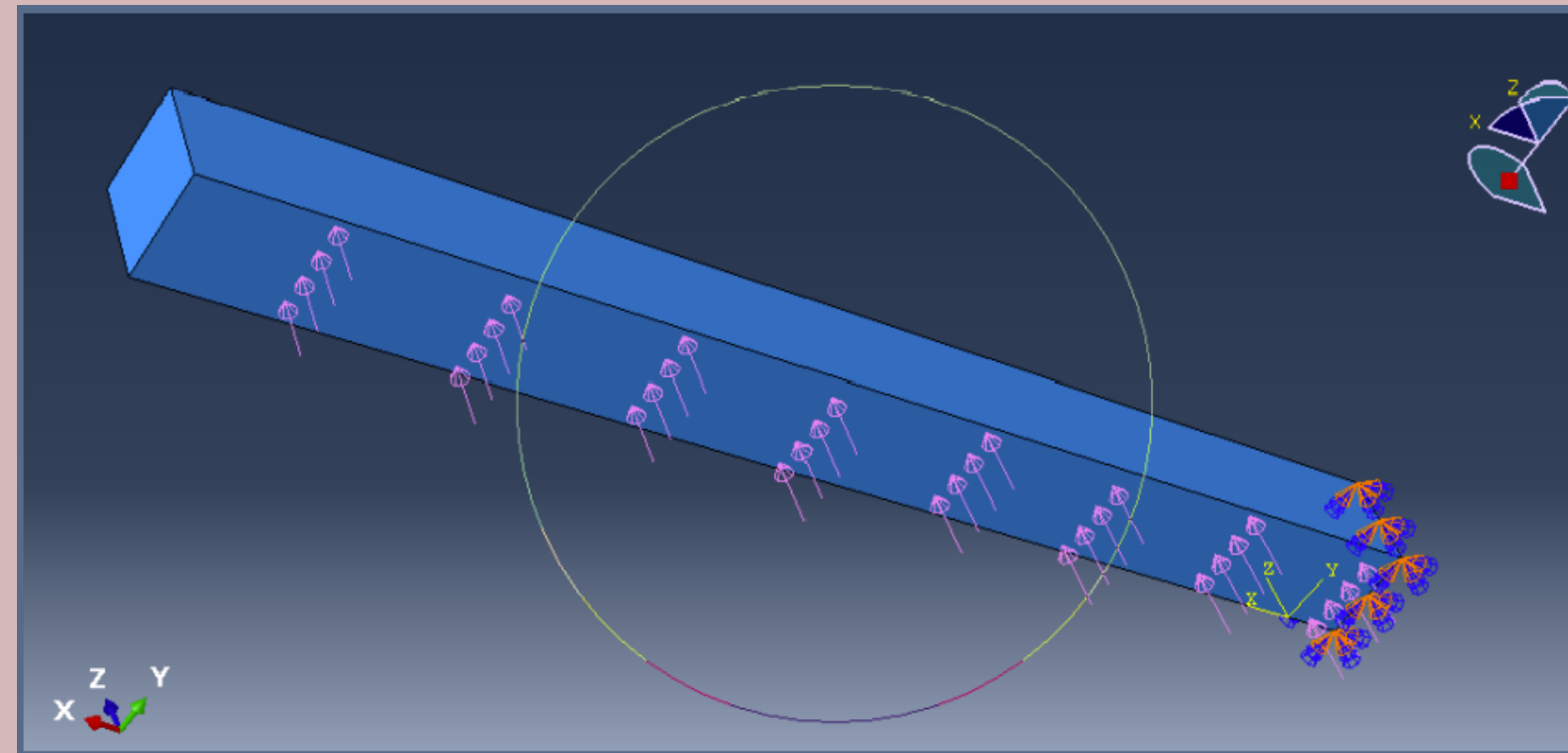


# BOUNDARY CONDITIONS

---

we consider the arm to be a clamped beam.

the beam is clamped, the deflection is zero



Pressure load

# BEAM ANALYSIS

On the y-axis we have:

$$+\uparrow \sum F_y = 0 \rightarrow F - R_{ay} = 0$$

$$F = R_{ay} = \underline{15N}$$

For the moments we have:

$$\sum M_z = 0 \rightarrow F \times L - M_{az} = 0$$

$$M_{az} = F \times L = 15 \times 0.25$$

$$M_{az} = \underline{3.75N.m}$$

To find the shear force (V) and the moment (M) we need to cut the beam.

On the y-axis we have:

$$+\uparrow \sum F_y = 0 \rightarrow V - R_{ay} = 0$$

$$V = R_{ay} = 15N$$

For the moment we have:

$$\sum M_z = 0 \rightarrow M_{az} - M_x - V \times x = 0$$

$$M_x = M_{az} - V \times x$$

$$M_x = 3.75 - 15x$$

At the support:

$$x = 0 \rightarrow M = 3.75N.m$$

For the slope we know that:

$$\theta = \int \frac{M}{EI} dx$$

Because EI is constant we get:

$$\theta = \frac{1}{EI} \int M dx$$

$$\theta = \frac{1}{EI} \int (3.75 - 15x) dx$$

$$\theta = \frac{1}{EI} \left( 3.75x - \frac{15}{2}x^2 + c_1 \right)$$

At the support:

$$\rightarrow \theta = 0$$

$$\theta = \frac{1}{EI} \left( 3.75x - \frac{15}{2}x^2 + c_1 \right) = 0$$

We replace  $x=0$  :

$$\theta = \frac{1}{EI} \left( 3.75 \times 0 - \frac{15}{2} \times 0^2 + c_1 \right) = 0$$

$$c_1 = 0$$

The final equation of the slope is:

$$\theta = \frac{1}{EI} \left( 3.75x - \frac{15}{2}x^2 \right)$$

For the deflection we know that:

$$\vartheta = \iint \frac{M}{EI} dx$$

Because EI is constant we get:

$$\theta = \frac{1}{EI} \iint M dx$$

$$\theta = \frac{1}{EI} \iint (3.75 - 15x) dx$$

$$\theta = \frac{1}{EI} \int \left( 3.75x - \frac{15}{2}x^2 + c_1 \right) dx$$

$$\theta = \frac{1}{EI} \left( \frac{3.75}{2}x^2 - \frac{15}{6}x^3 + c_1x + c_2 \right)$$

We deduced previously that  $c_1 = 0$  so:

$$\theta = \frac{1}{EI} \left( \frac{3.75}{2}x^2 - \frac{15}{6}x^3 + c_2 \right)$$

At the support  $\theta = 0$  :

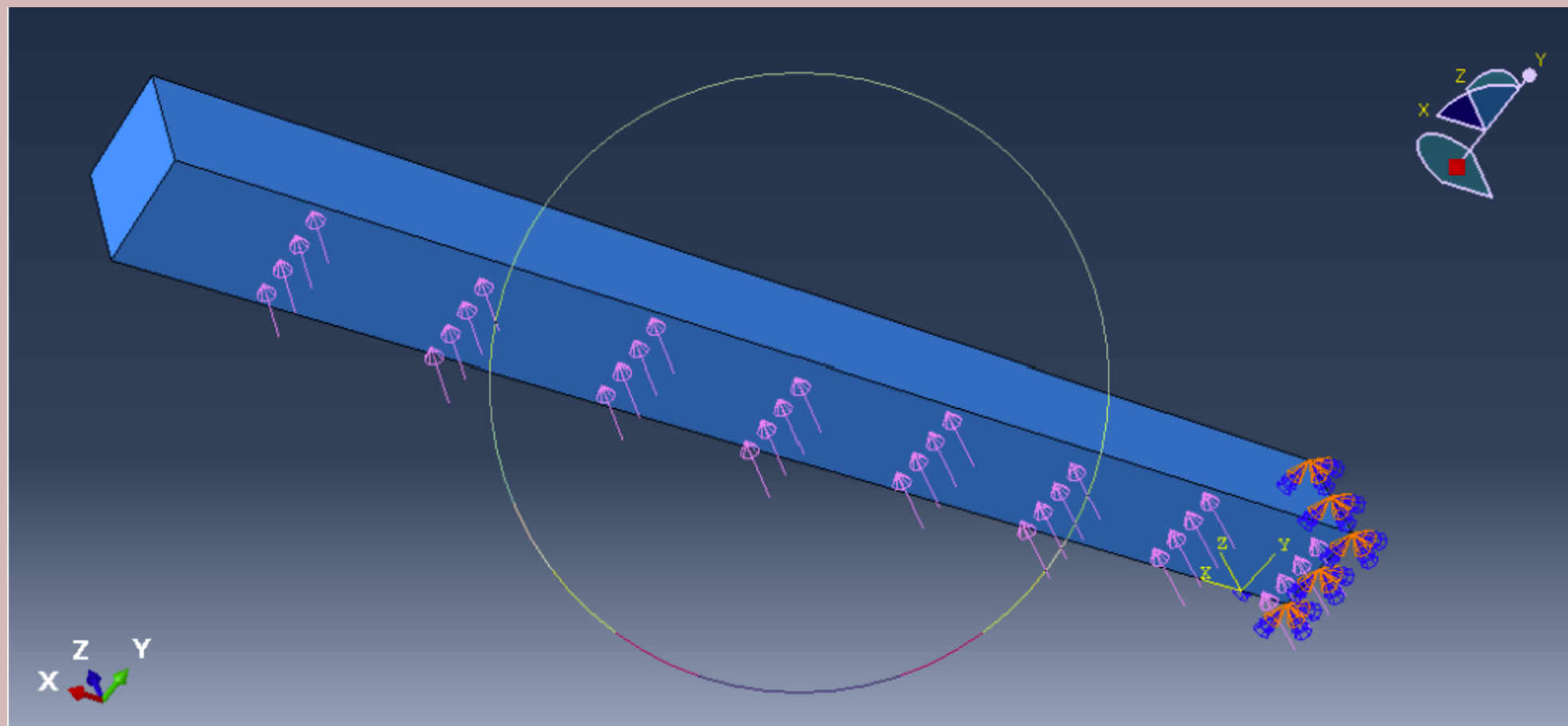
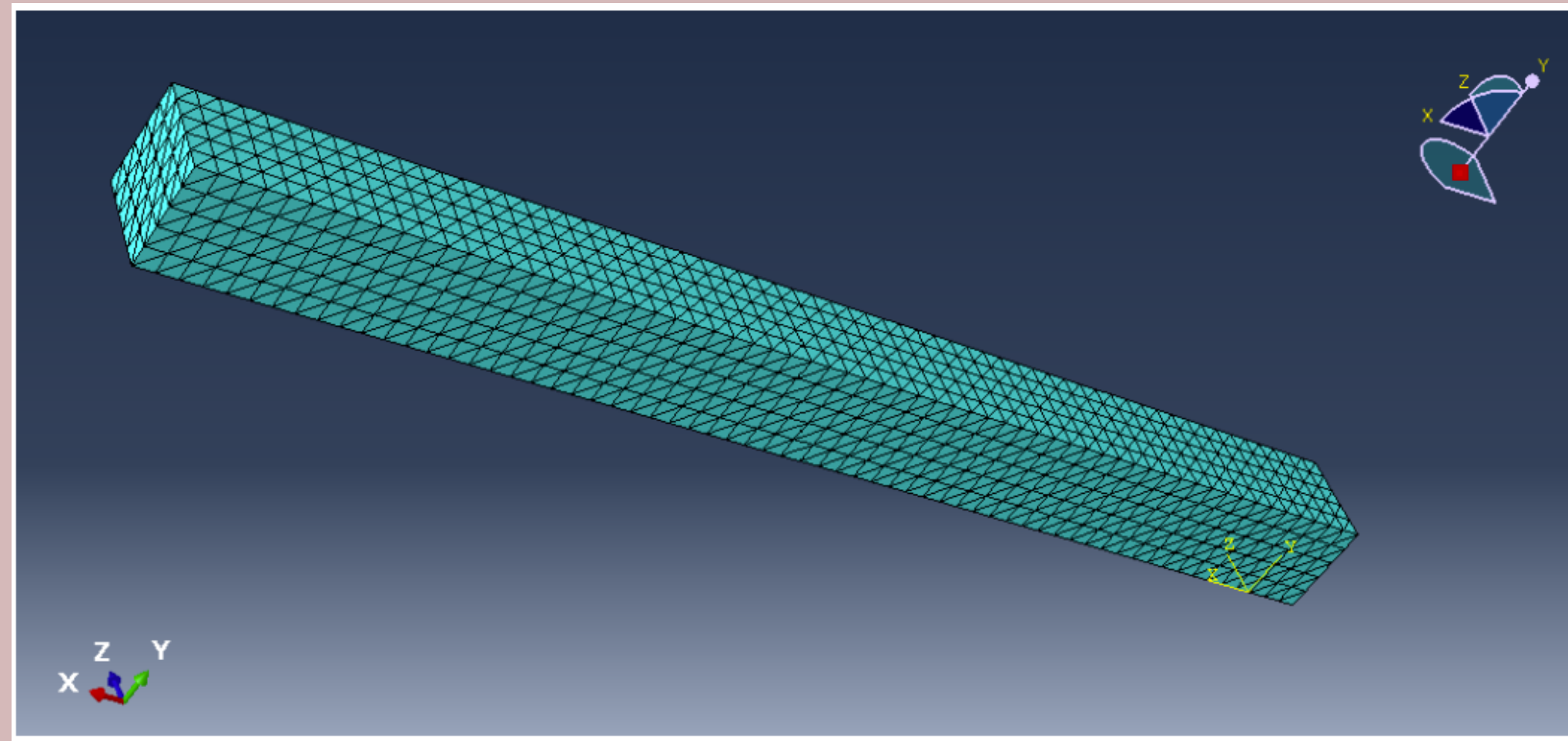
$$\theta = \frac{1}{EI} \left( \frac{3.75}{2}x^2 - \frac{15}{6}x^3 + c_2 \right) = 0$$

We replace  $x=0$ :

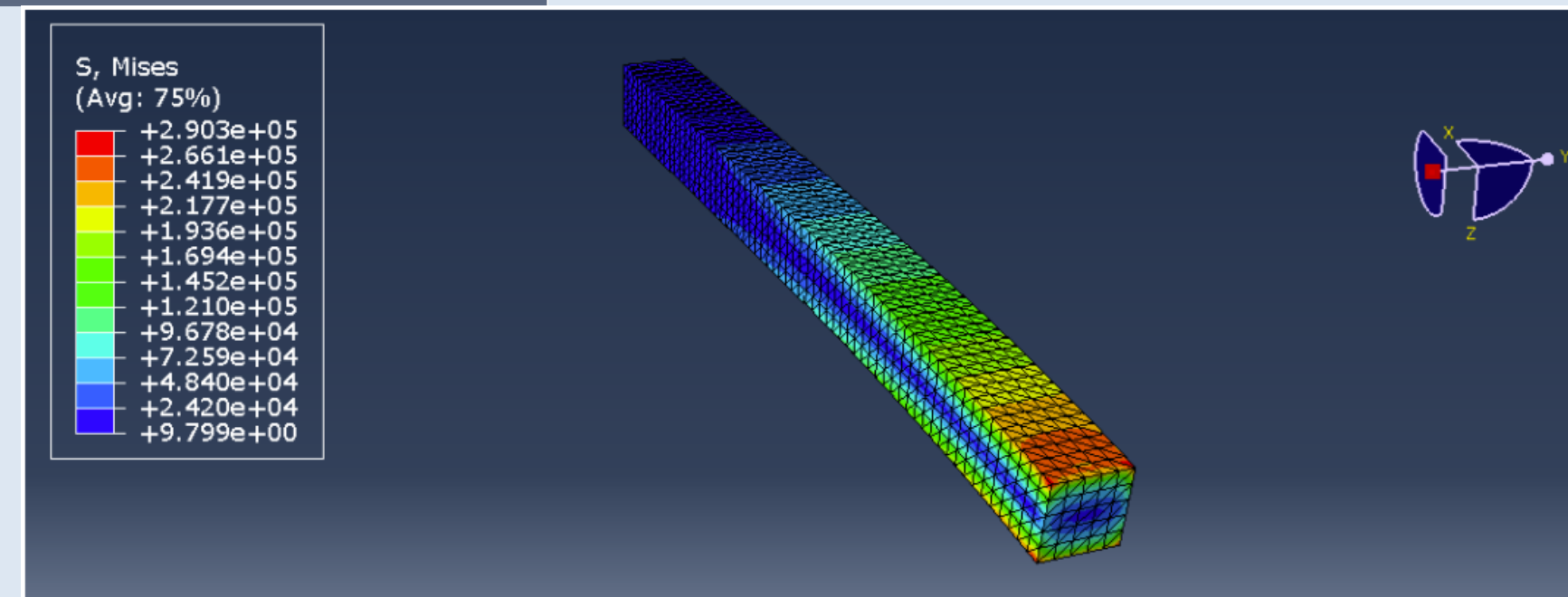
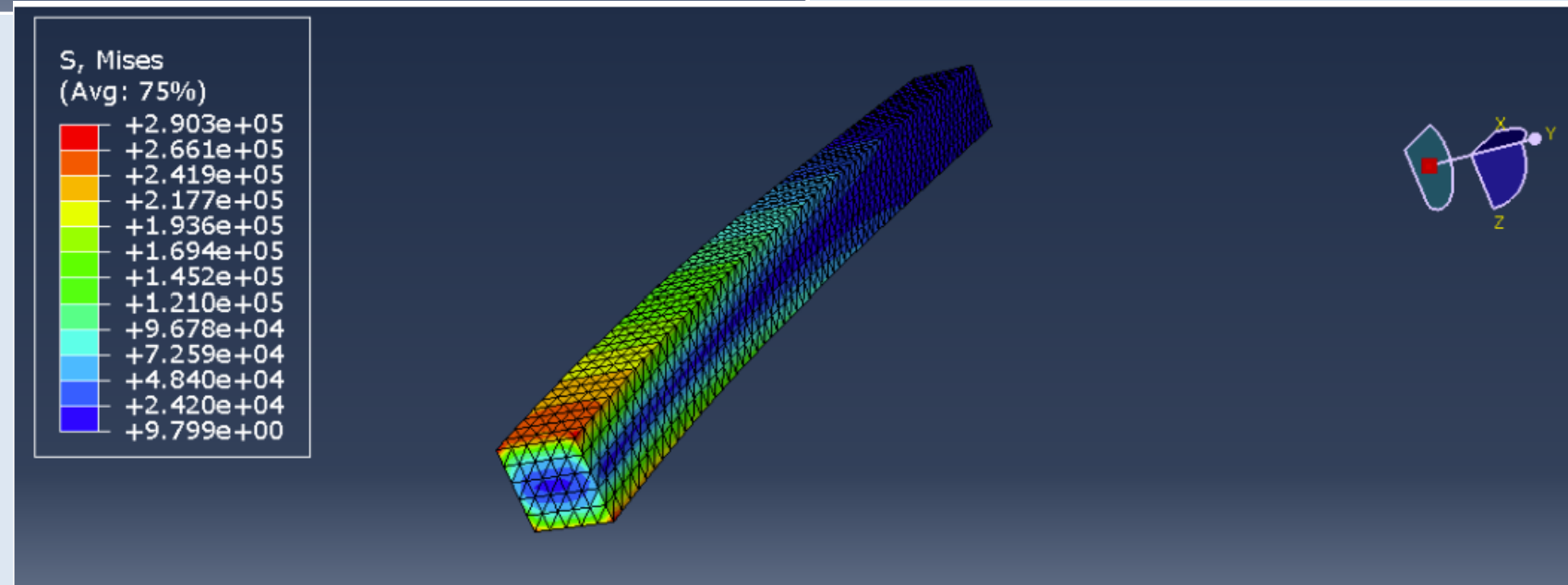
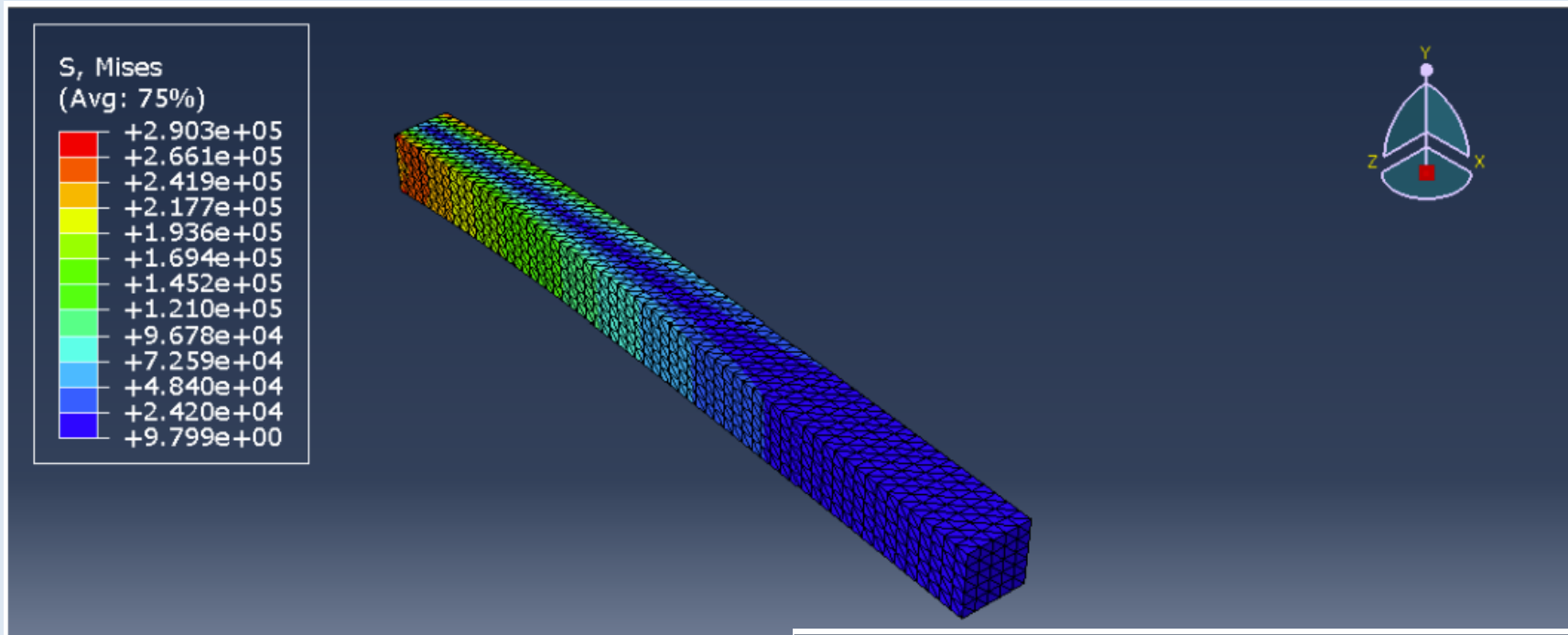
$$\frac{1}{EI} \left( \frac{3.75}{2}x 0^2 - \frac{15}{6}x 0^3 + c_2 \right) = 0$$
$$c_2 = 0$$

The final equation of the deflection is:

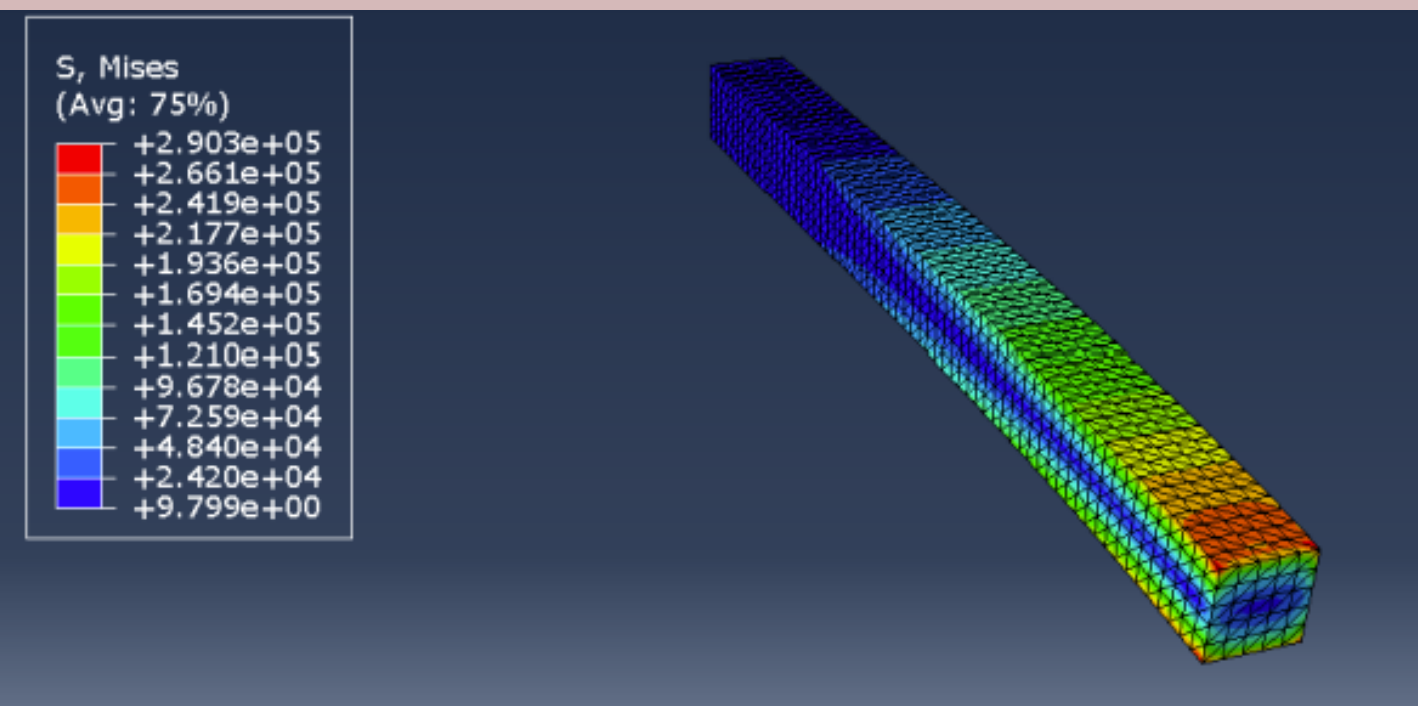
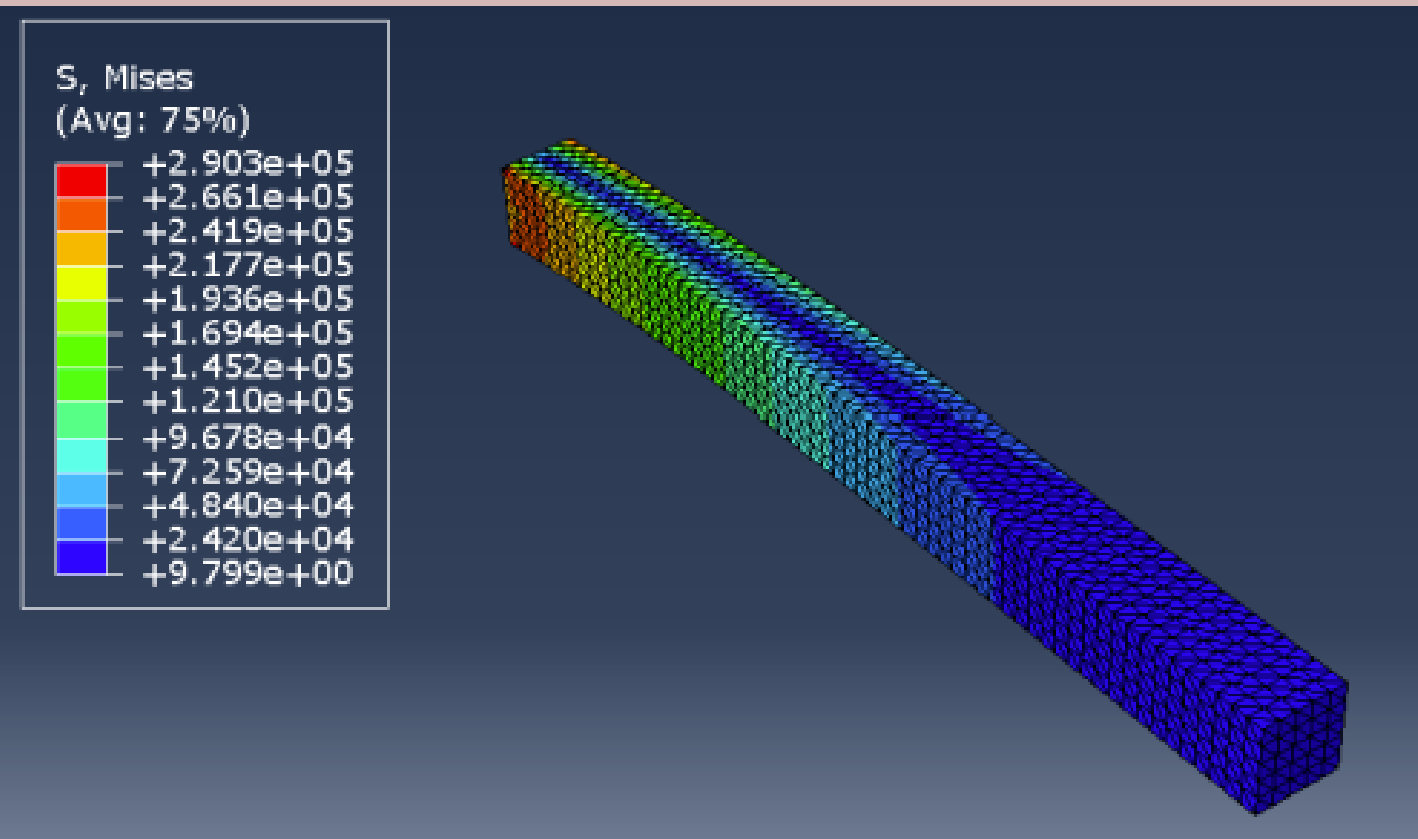
$$\theta = \frac{1}{EI} \left( \frac{3.75}{2}x^2 - \frac{15}{6}x^3 \right)$$



# SIMULATIONS



# RESULTS AND DISCUSSION



- the stress concentration is higher when getting close to the fixed part.
- the stress concentration is cancelled in the neutral axis
- the clamped part should be stronger and well built so as to avoid any failure.



**THANK YOU FOR  
YOUR ATTENTION**