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# Finite Element Models for Sound Transmission through Foam Wedges and Foam Layers Having Spatially Graded Properties

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**Finite Element Models for Sound Transmission  
through Foam Wedges and Foam Layers  
Having Spatially Graded Properties**

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Automated Analysis Corporation**

**The 130th Meeting of the Acoustical Society of America  
St. Louis, MO  
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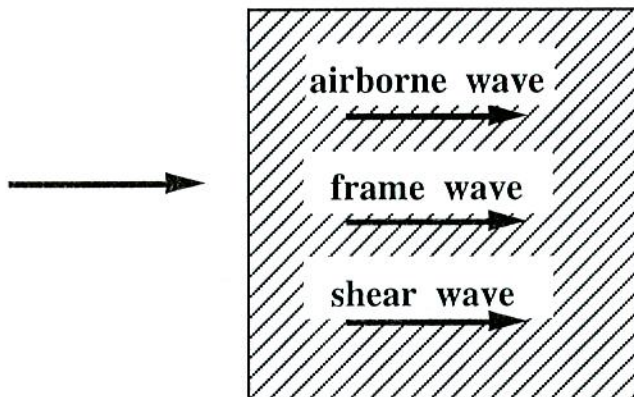
**PURDUE UNIVERSITY**



**HERRICK LABORATORIES**

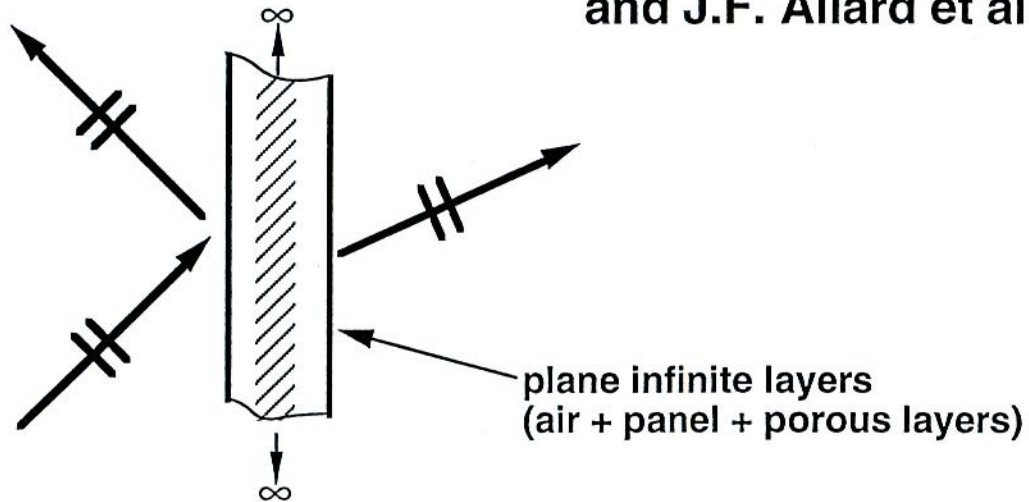
# ELASTIC POROUS MATERIALS

- *Foam Material Properties*
  - Flow resistivity
  - Tortuosity (Structure Factor)
  - Porosity
  - Bulk Modulus of Elasticity
  - Poisson's Ratio
  - Loss Factor
- Biot Theory allows wave propagation to be expressed in terms of these macroscopically measurable properties.
- Multiple Wave Types → sensitive to boundary conditions

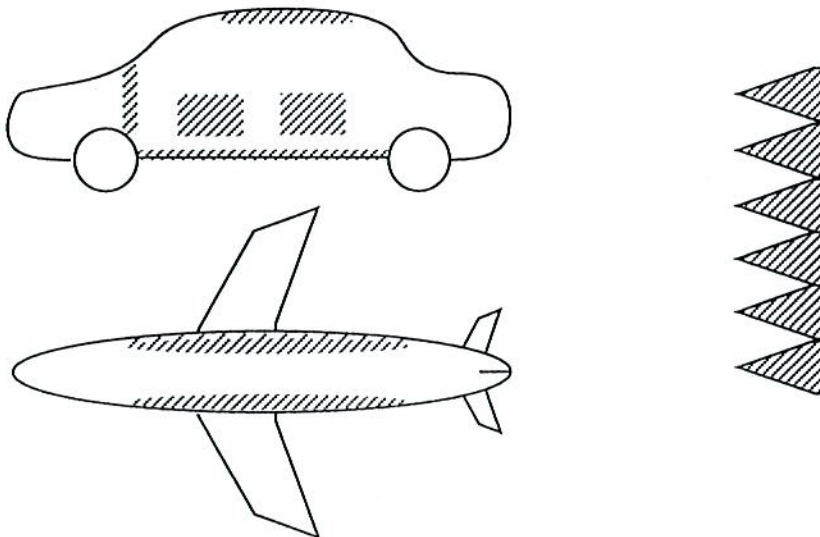


# INTRODUCTION

- **Analytical Capabilities** — available by J.S. Bolton and J.F. Allard et al.

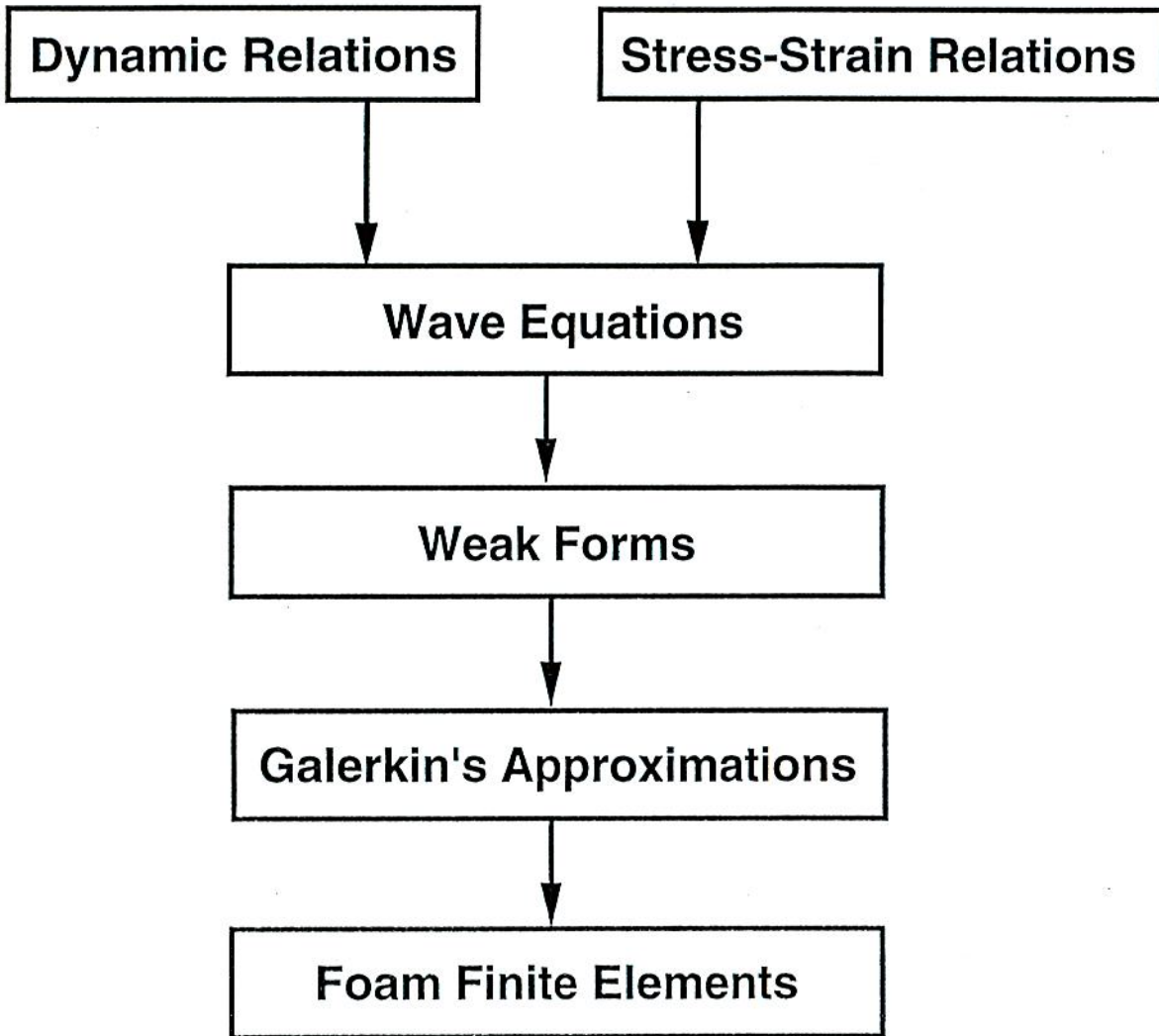


- **Practical Treatments**



# FOAM FINITE ELEMENTS

*Elastic Porous Material Theory based on Biot*



# Finite Element Models/Foam Properties

- **FEM Models**

- (i) **Foam**

- 3-noded linear triangular elements

- (ii) **Structure**

- 2-noded hermite cubic elements

- (iii) **Airspace**

- 3-noded linear triangular elements

- **Foam Properties**

- bulk density of solid phase:  $30 \text{ kg/m}^3$

- bulk Young's modulus:  $8 \times 10^5 (1 + j 0.265) \text{ Pa}$

- bulk Poisson's ratio: 0.4

- flow resistivity: 25,000 MKS Rayls/m

- tortuosity: 7.8

- porosity: 0.9



# BOUNDARY CONDITIONS

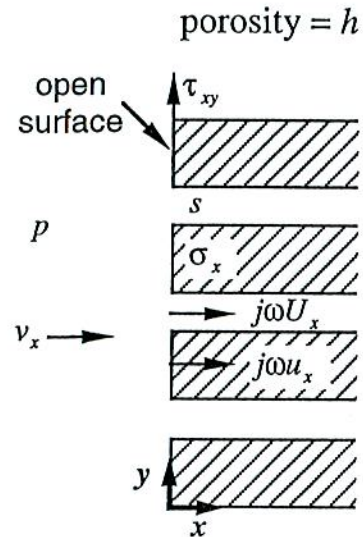
## ● OPEN SURFACE

$$-hp = s$$

$$-(1-h)p = \sigma_x$$

$$\tau_{xy} = 0$$

$$v_x = j\omega(1-h)u_x + j\omega hU_x$$



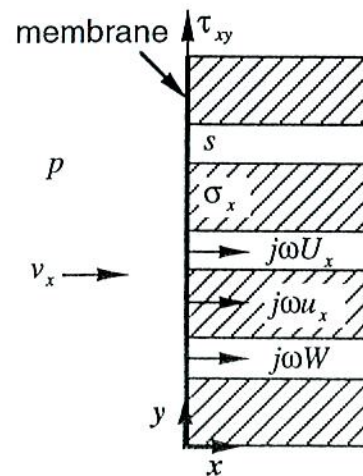
## ● MEMBRANE-SEALED SURFACE

$$v_x = j\omega W$$

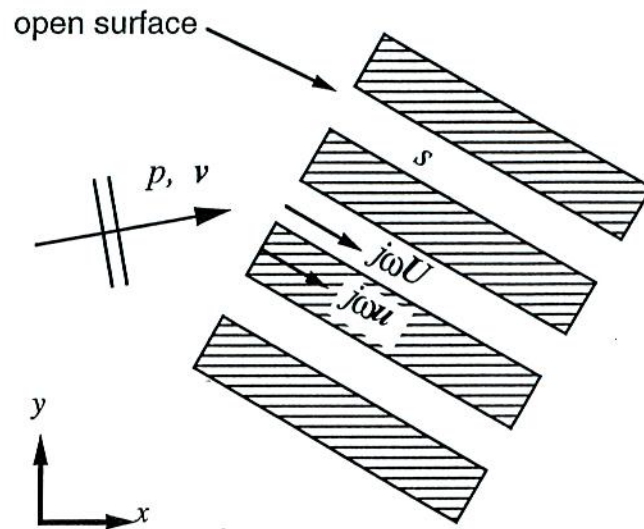
$$u_x = W$$

$$U_x = W$$

$$\pm p \pm (\sigma_x + s) = -\omega^2 m_s W$$



## BOUNDARY CONDITIONS — OPEN SURFACE



$$hp\mathbf{n}_a = s\mathbf{n}_f$$

$$(1-h)pn_{ax} = \sigma_x n_{fx} + \tau_{xy}n_{fy}$$

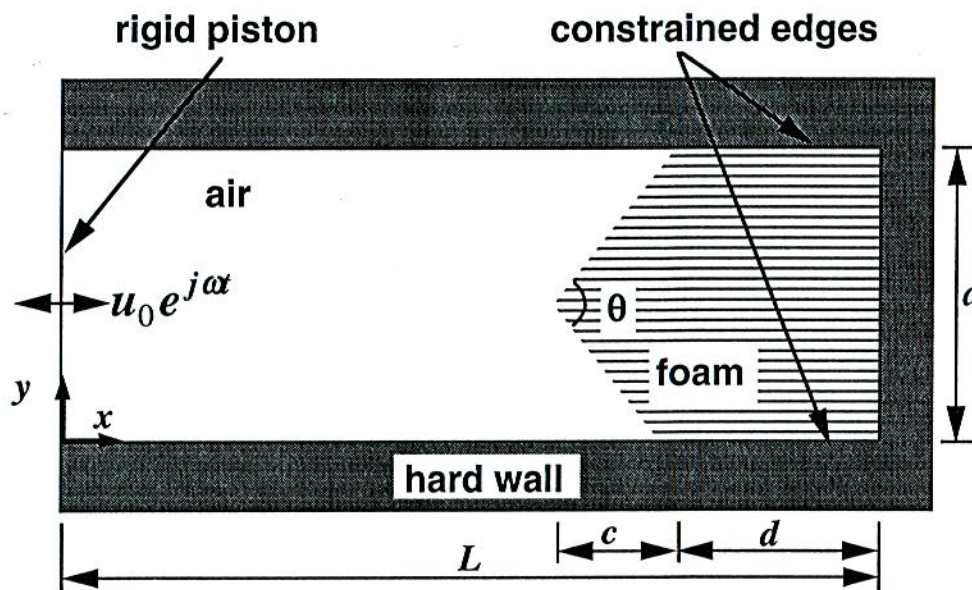
$$(1-h)pn_{ay} = \tau_{xy}n_{fx} + \sigma_y n_{fy}$$

$$\mathbf{v} = j\omega(1-h)\mathbf{u} + j\omega h\mathbf{U}$$



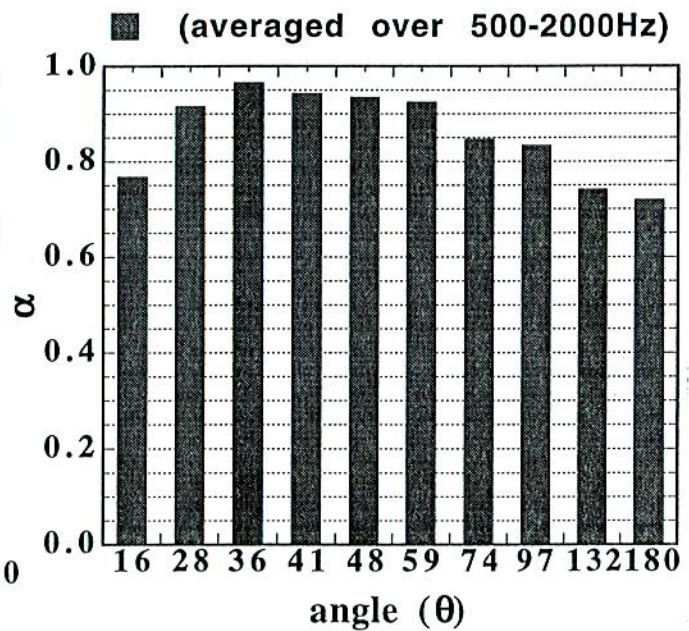
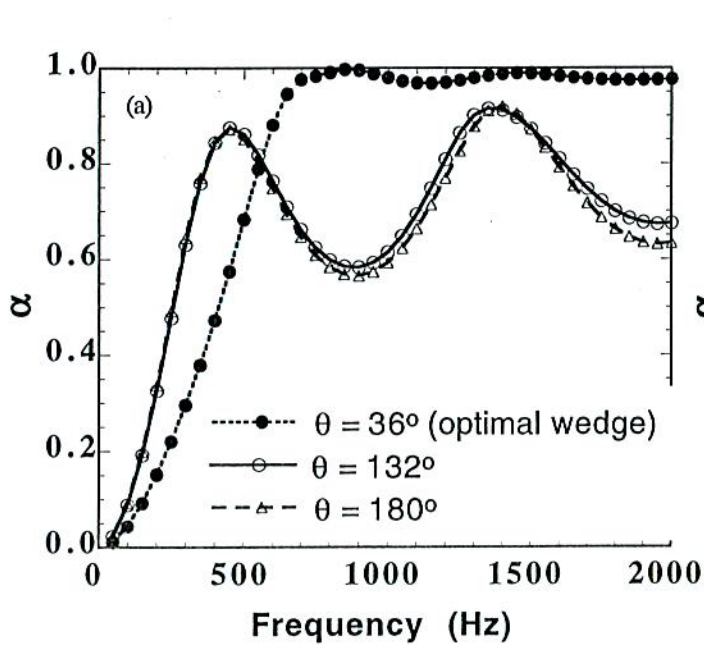
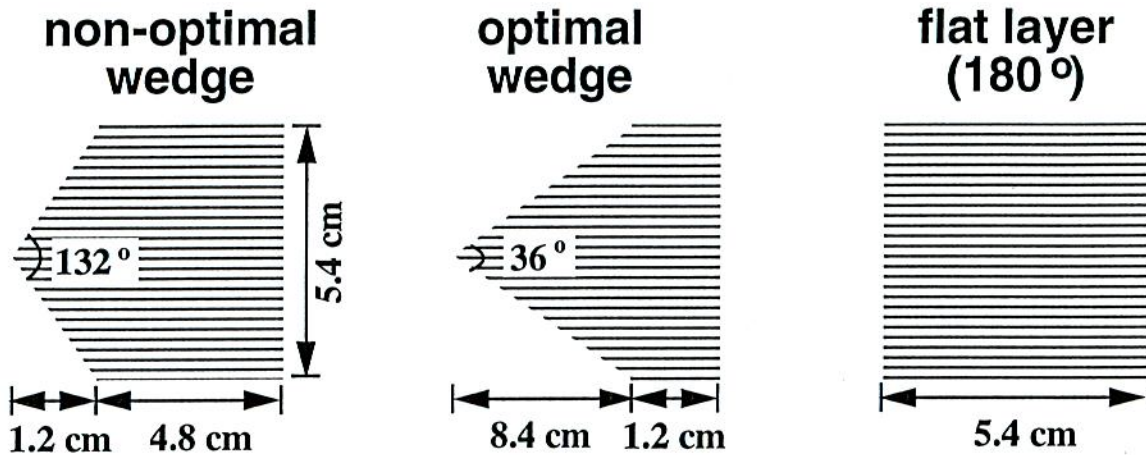
# SHAPE OPTIMIZATION OF FOAM WEDGE

- **Objective** – maximize absorption offered by a wedge over a specified frequency range



- wedge defined by  $\theta$  when volume and  $a$  is held constant
- given volume find optimum angle,  $\theta$

# SHAPE OPTIMIZATION OF FOAM WEDGE



# MOTIVATIONS AND OBJECTIVE

## ● Motivations

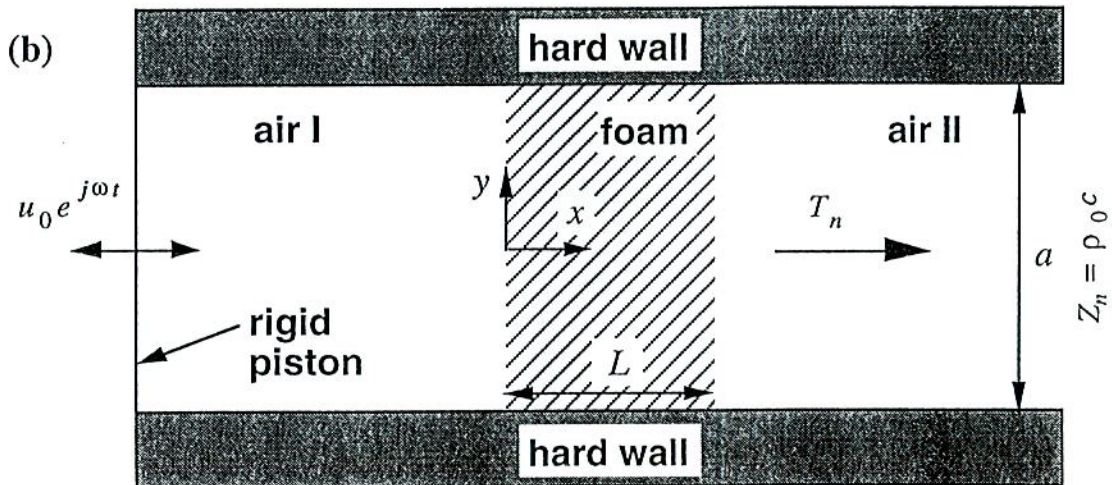
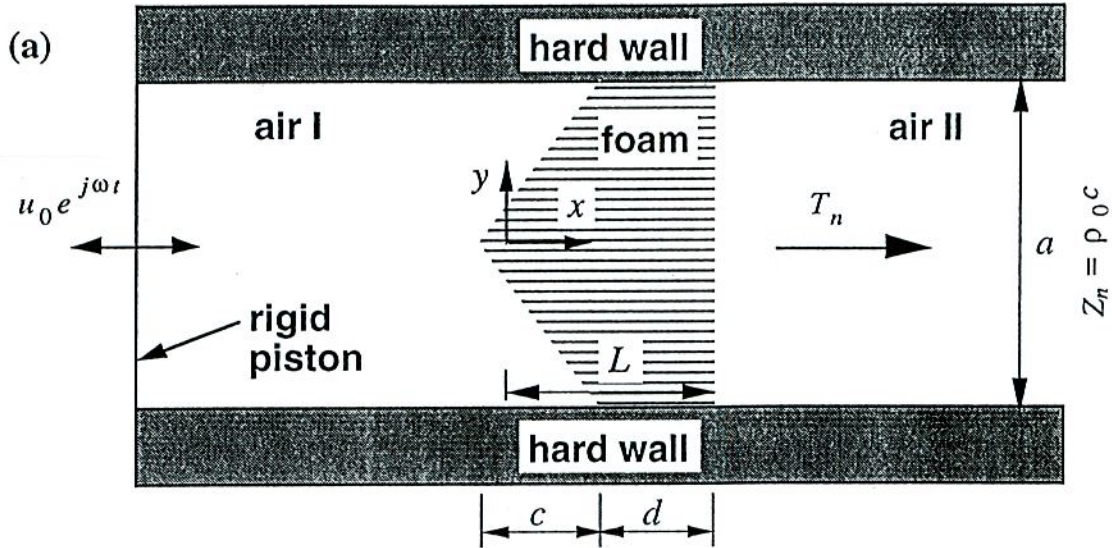
- could optimize shape of a foam wedge by maximizing absorption offered by a wedge over a specified frequency range
- found interesting and useful sound transmission characteristics of foam wedge in some high frequency bands
- required larger treatment spaces if foam wedges are used

## ● Objective

- to find a way of increasing sound transmission loss of a plane foam layer based on facts that were found from wedge studies

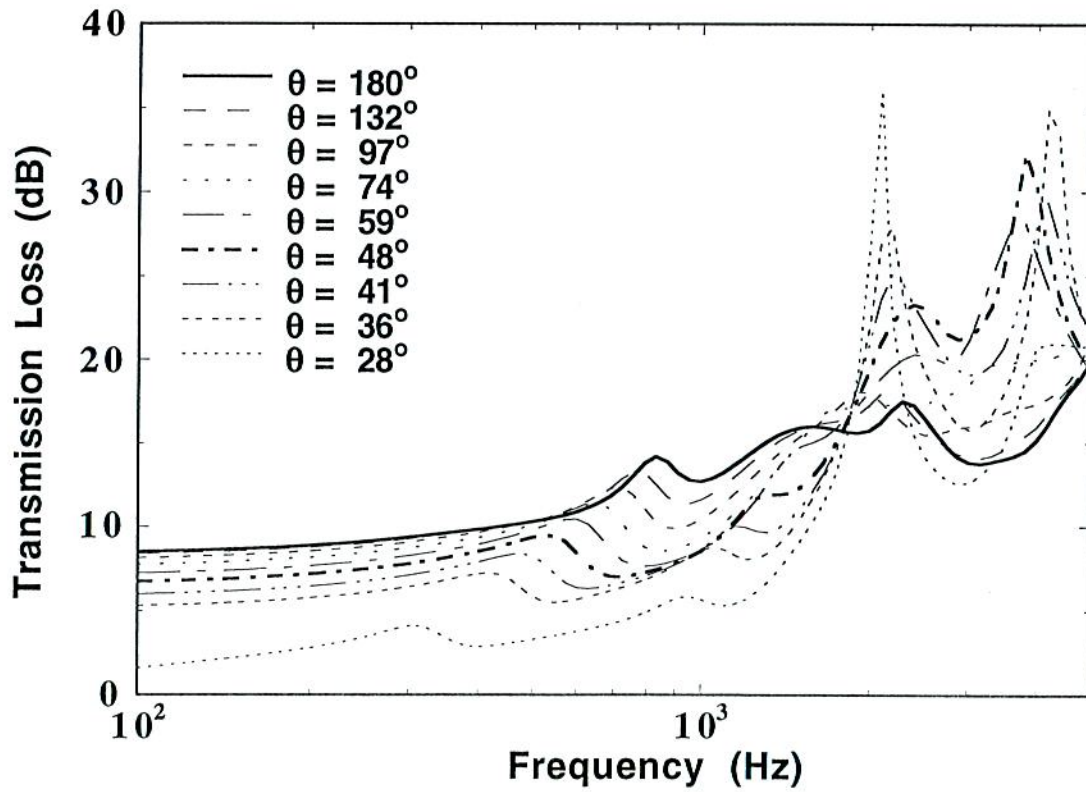
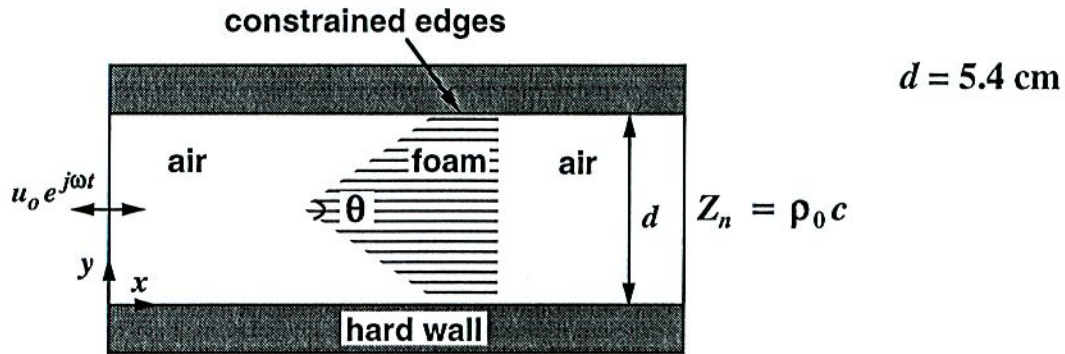


# SYSTEM CONFIGURATIONS

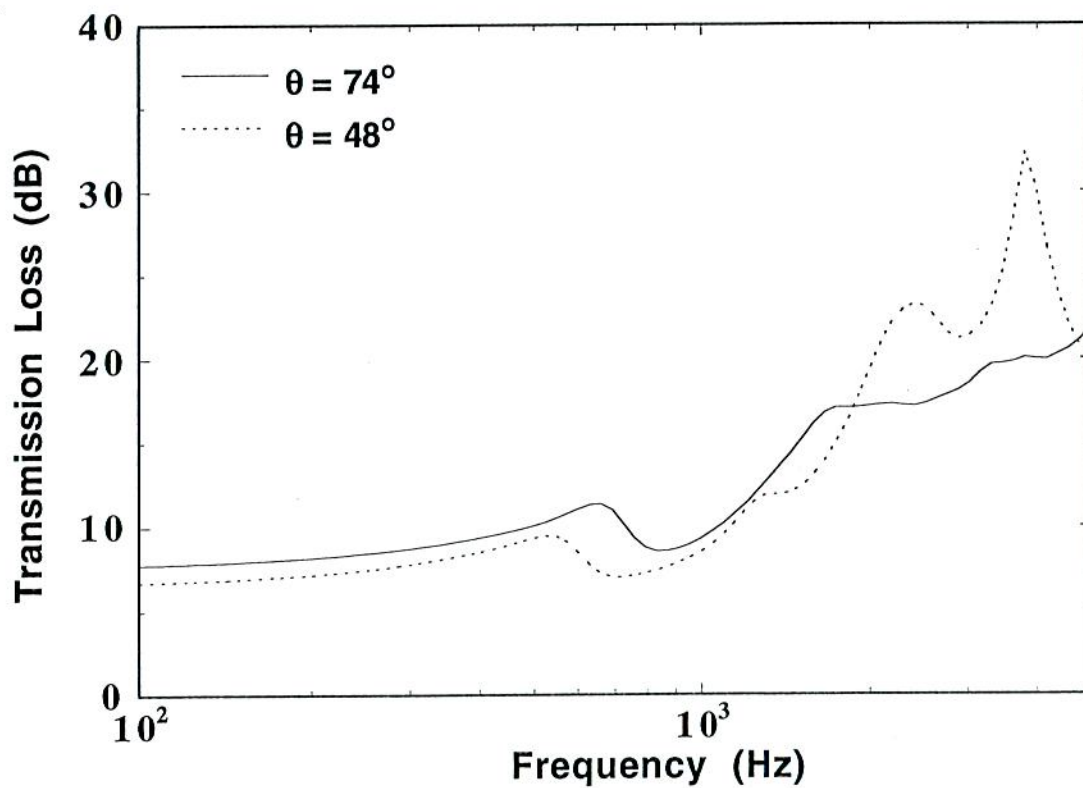
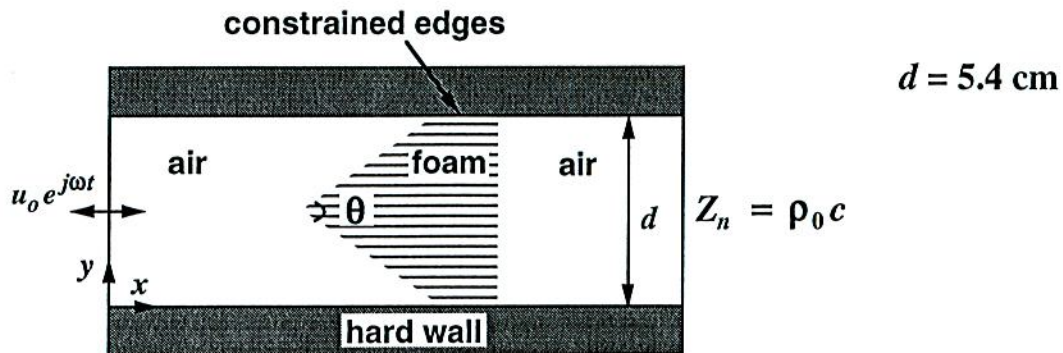


In system (b), tortuosity of a foam layer is varied spatially across the duct (in  $y$ -direction).

# SOUND TRANSMISSION THROUGH A WEDGE



# SOUND TRANSMISSION THROUGH A WEDGE



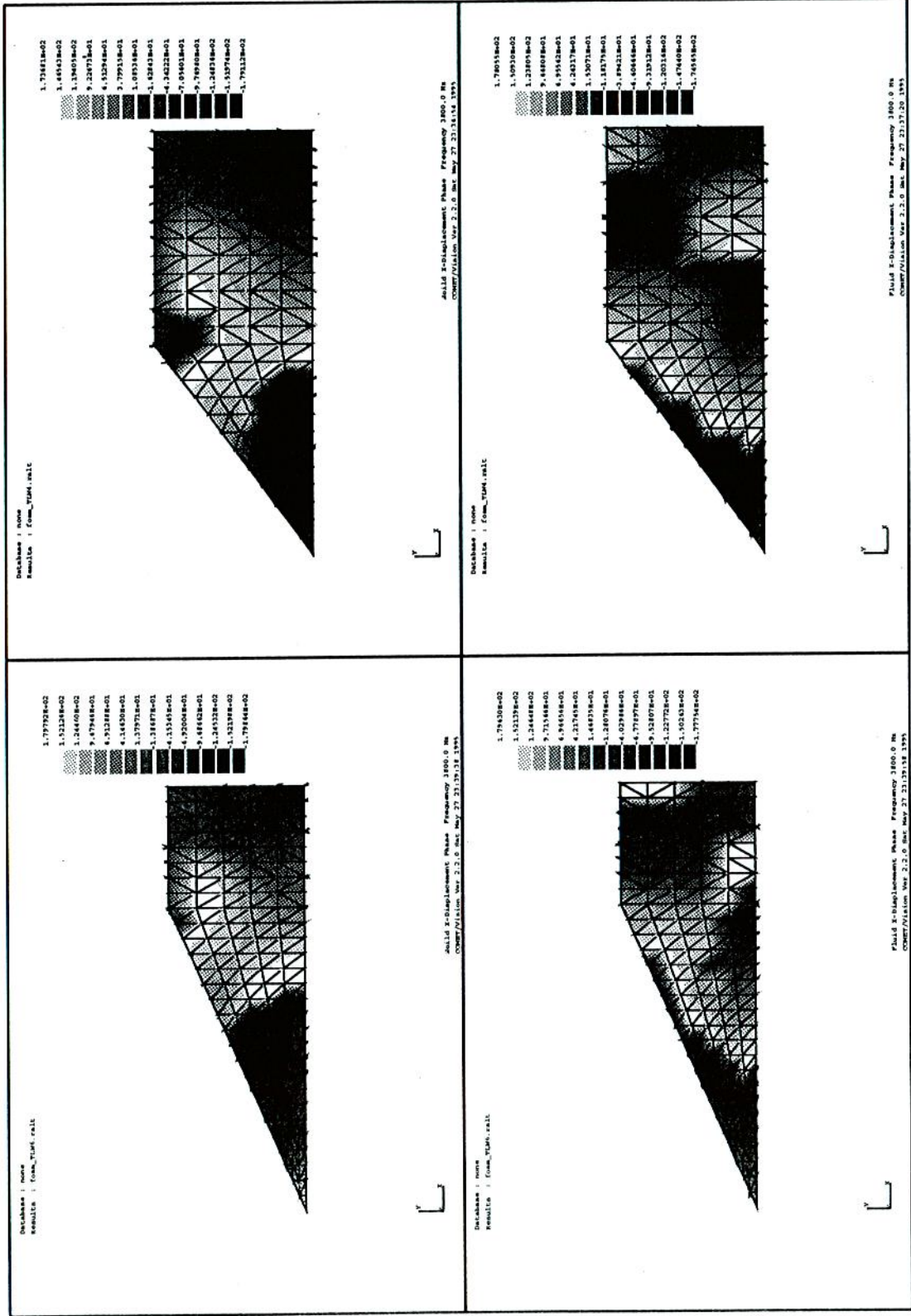
# PHASES OF THE X-DISPLACEMENTS OF THE SOLID AND FLUID PHASES (f = 3800 Hz)

$\theta = 48^\circ$

$\theta = 74^\circ$

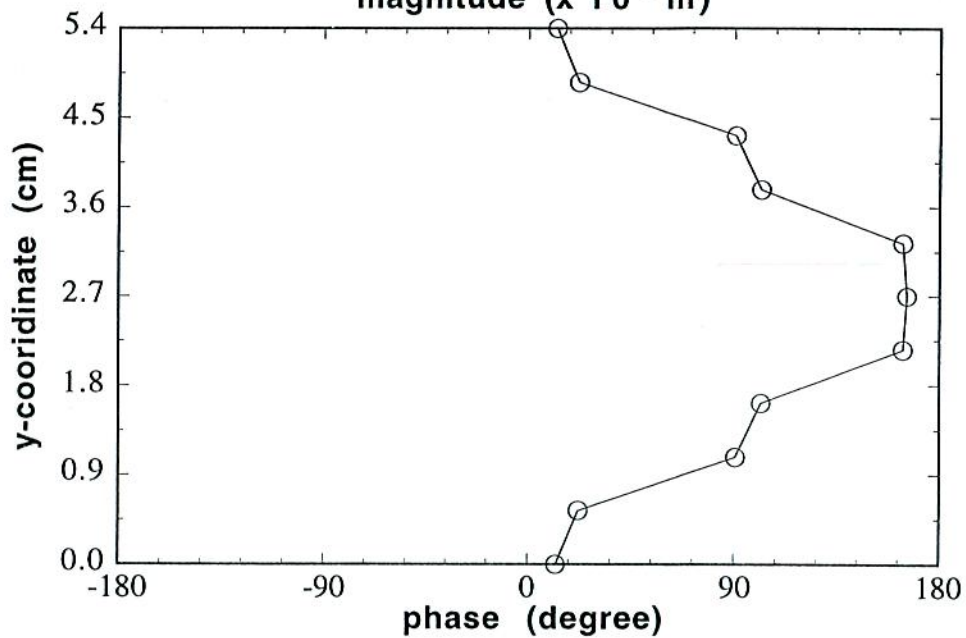
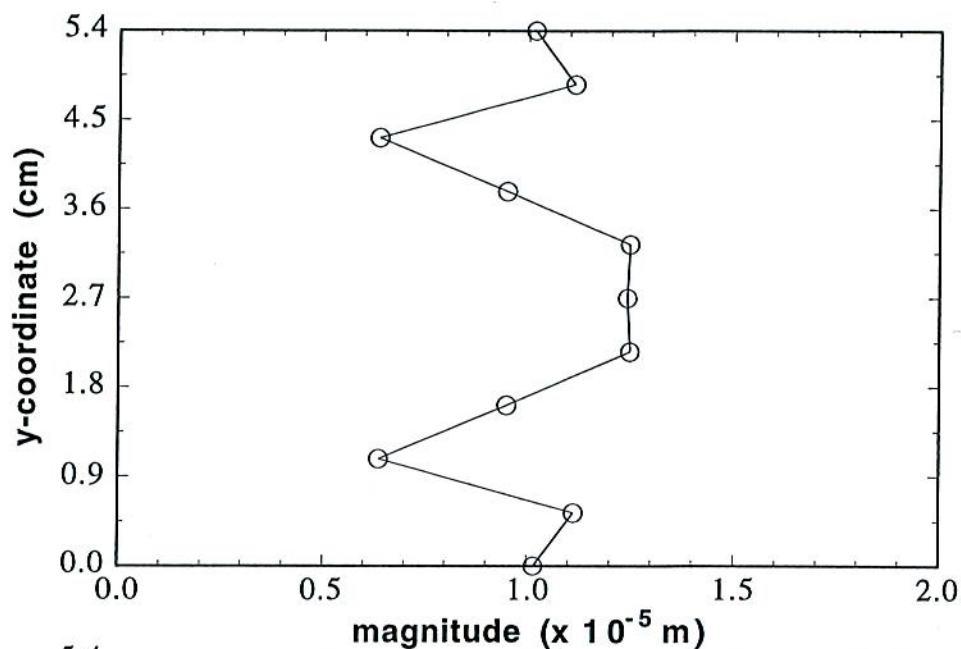
**Solid**

**Fluid**



# MAGNITUDE AND PHASE OF THE X-DISPLACEMENT OF THE FLUID PHASE AT THE REAR SURFACE OF THE WEDGE

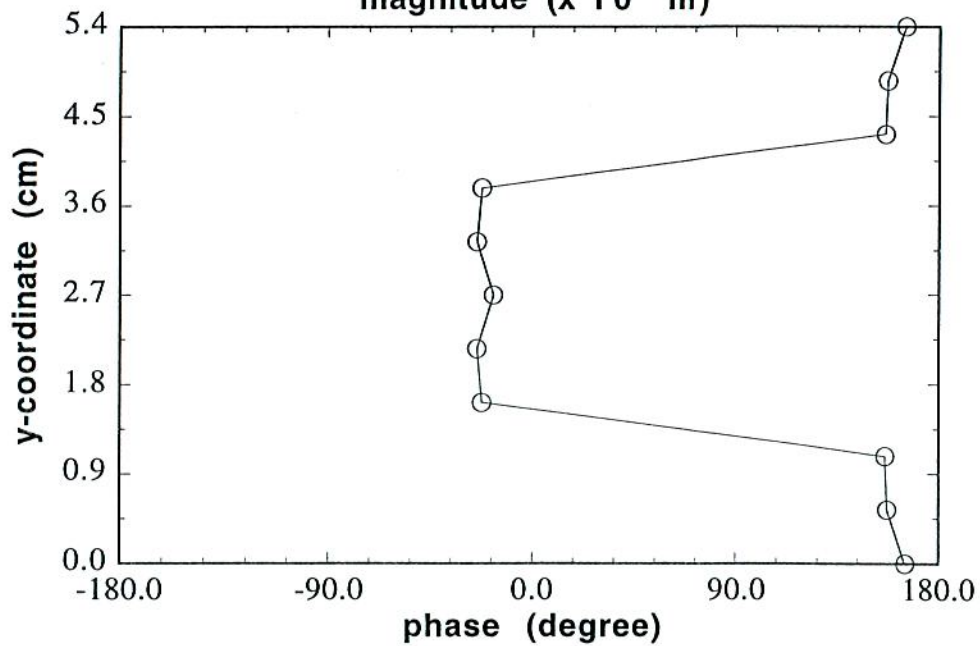
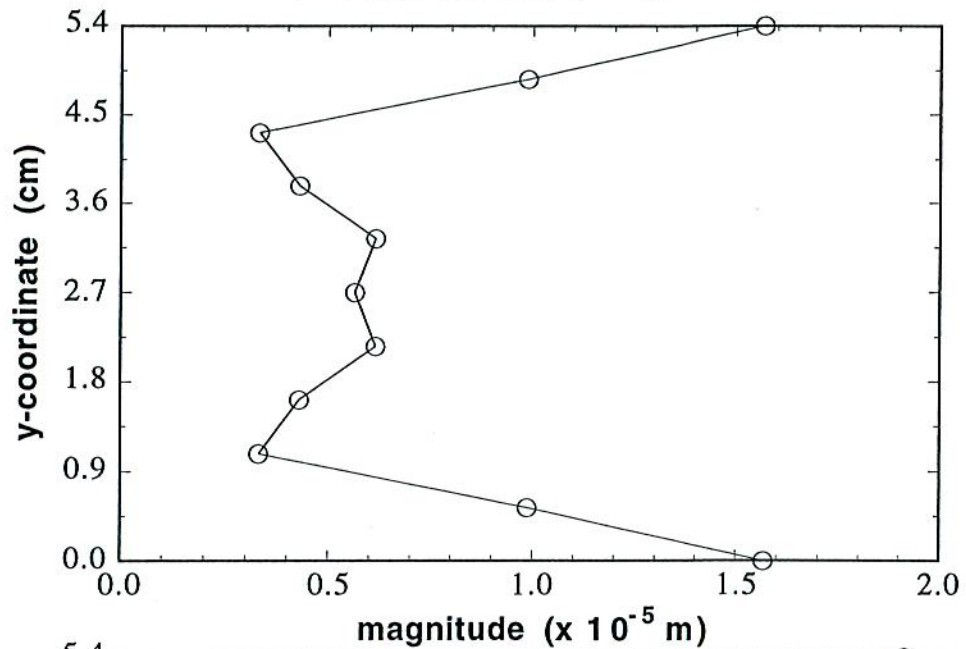
$f = 2300 \text{ Hz}$  and  $\theta = 48^\circ$



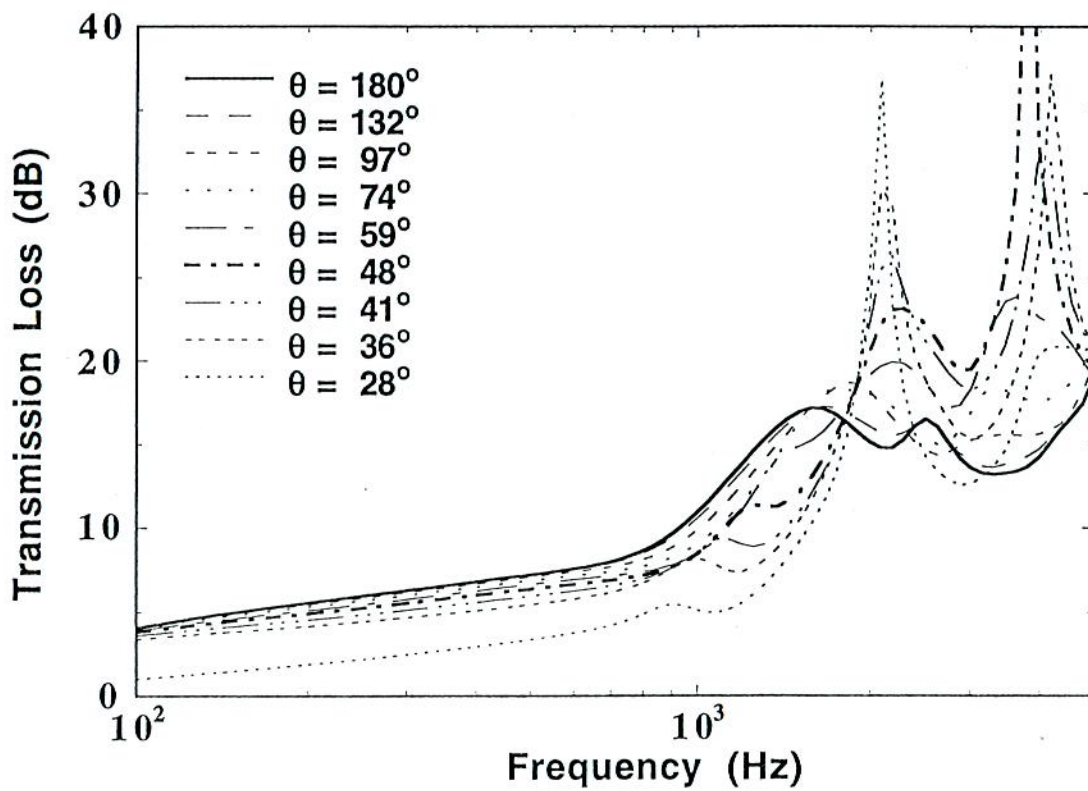
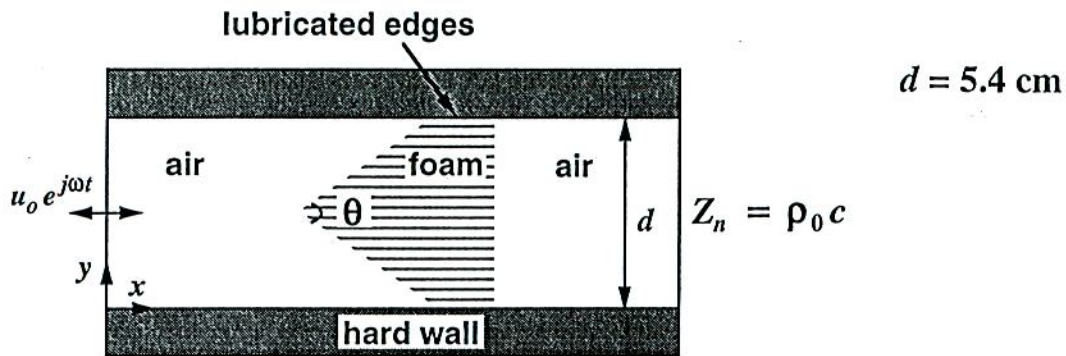


# MAGNITUDE AND PHASE OF THE X-DISPLACEMENT OF THE FLUID PHASE AT THE REAR SURFACE OF THE WEDGE

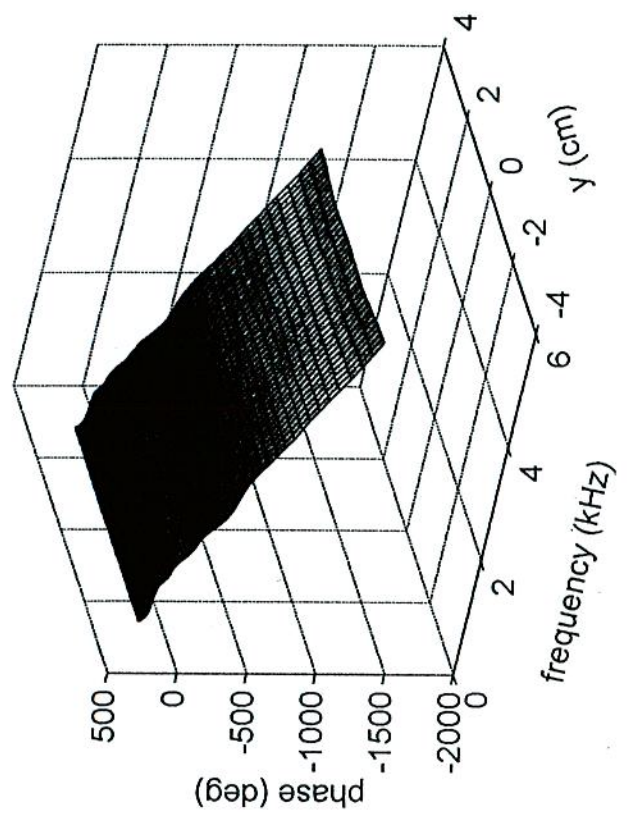
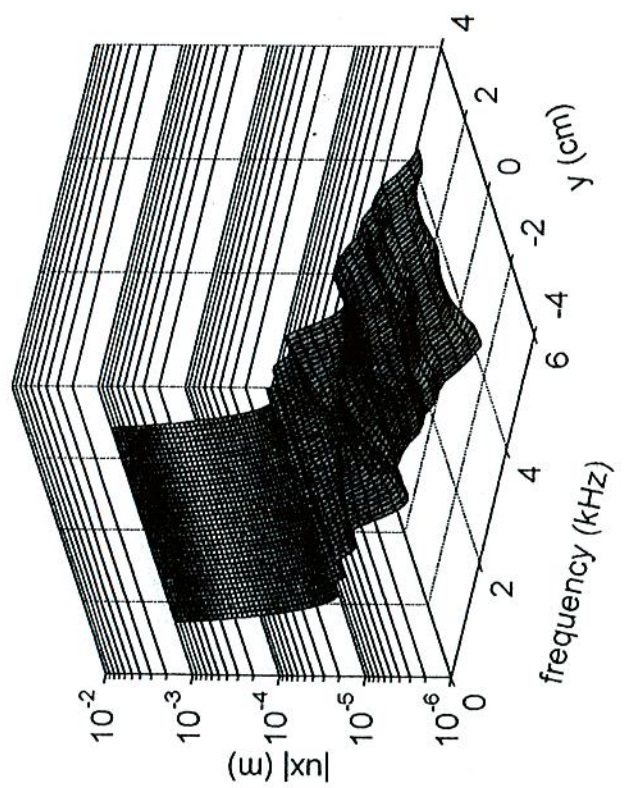
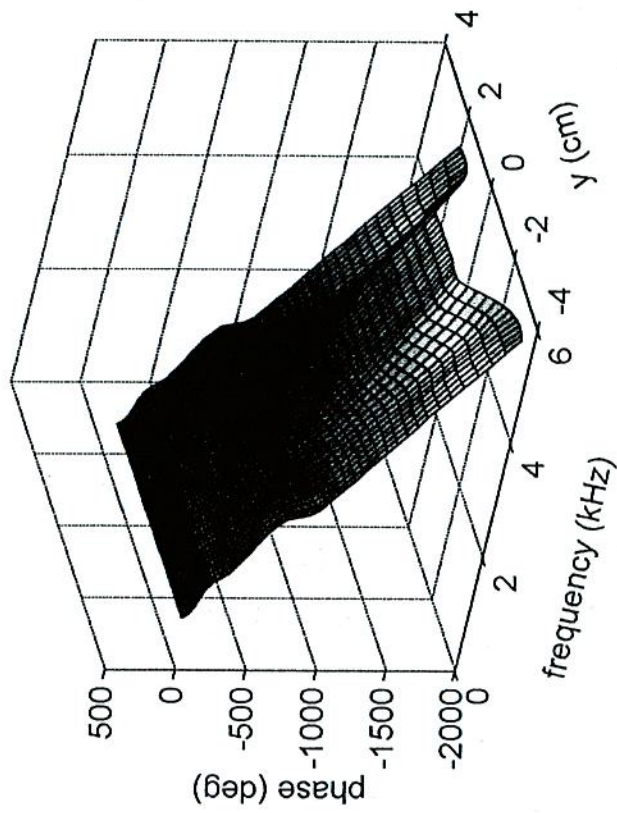
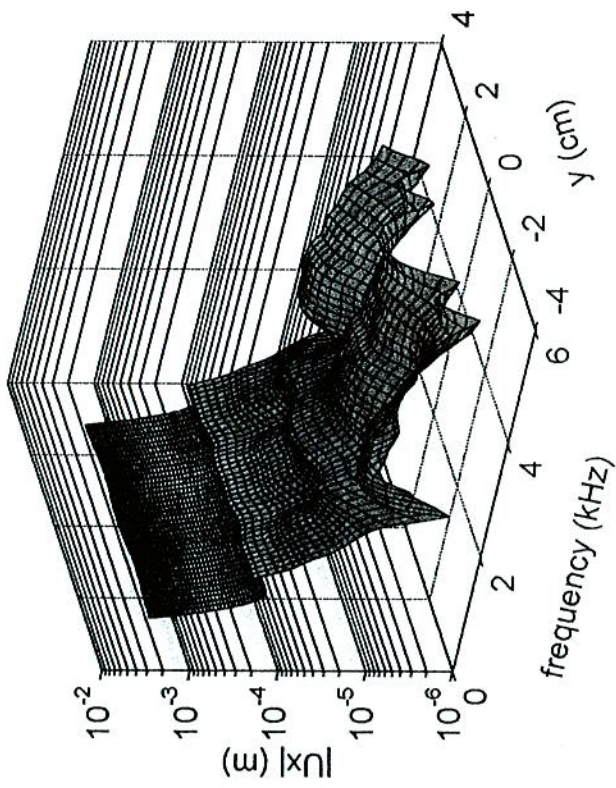
$f = 3800 \text{ Hz}$  and  $\theta = 48^\circ$



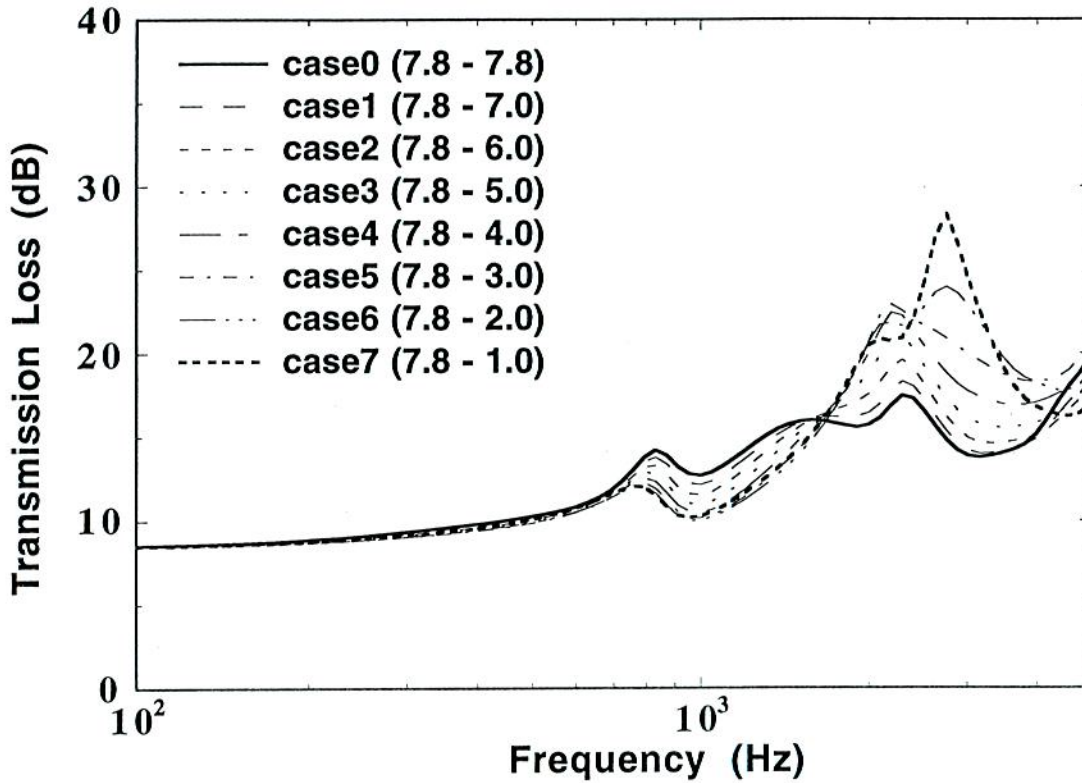
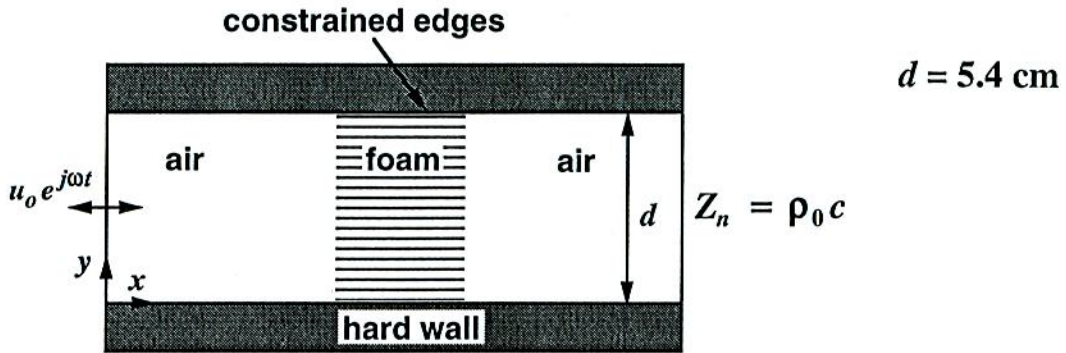
# SOUND TRANSMISSION THROUGH A WEDGE



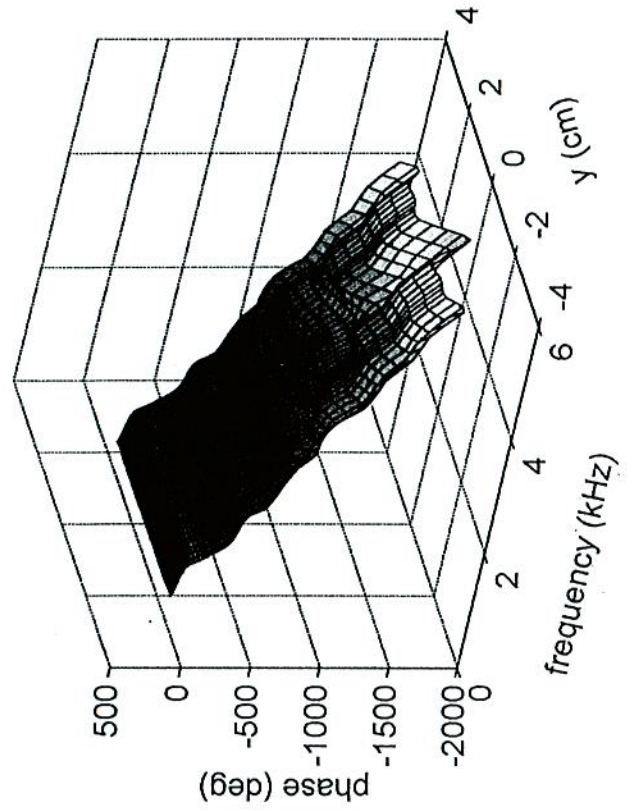
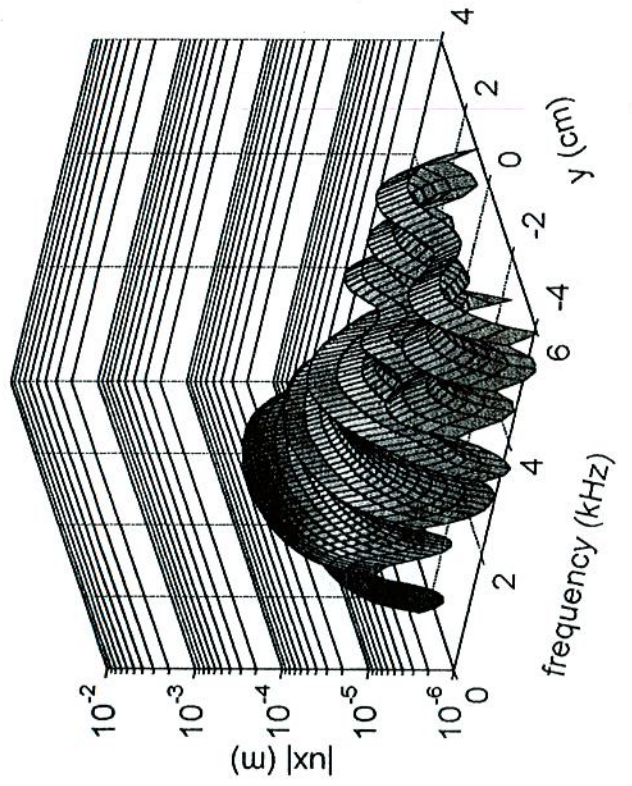
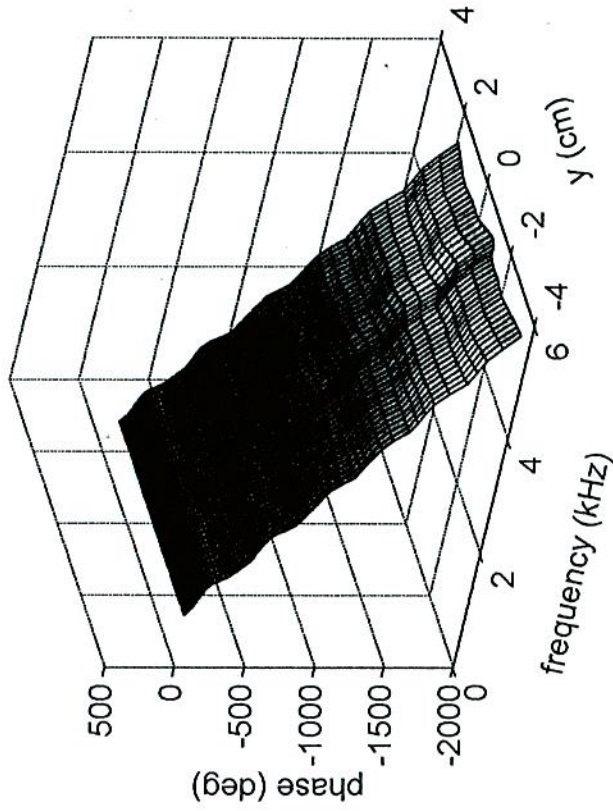
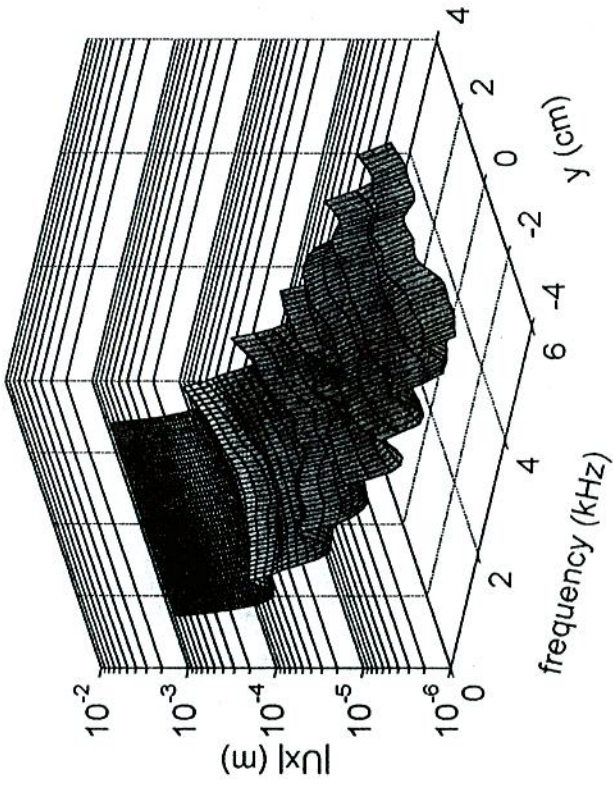
LUBRICATED FOAM WEDGE



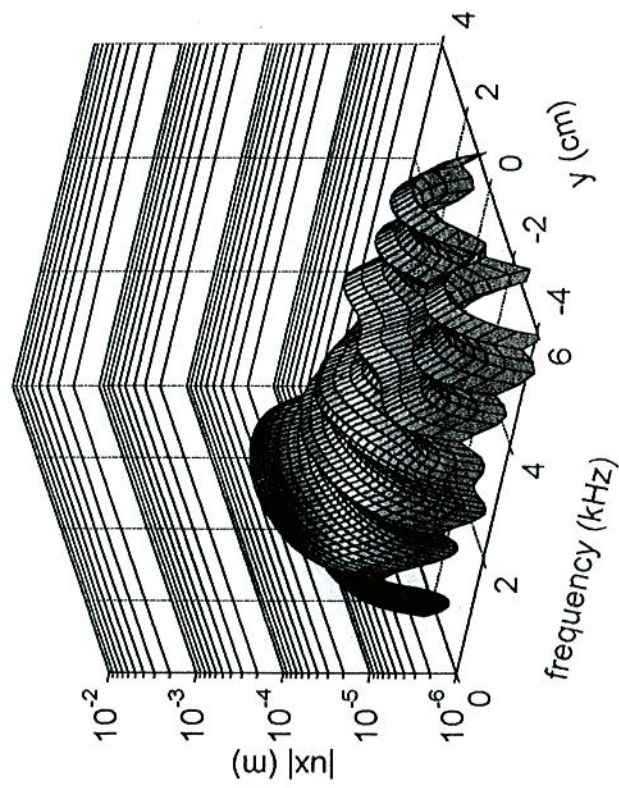
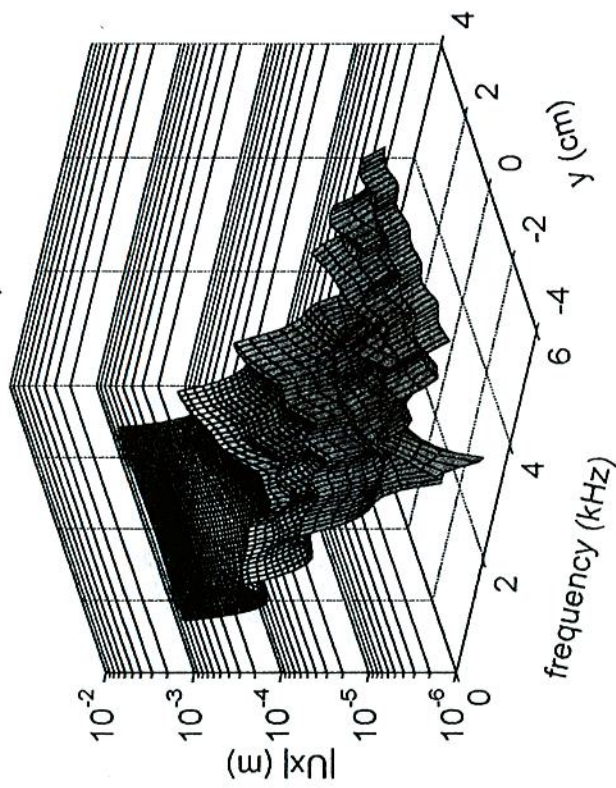
# SOUND TRANSMISSION THROUGH A FOAM LAYER HAVING SPATIALLY GRADED TORTUOSITY



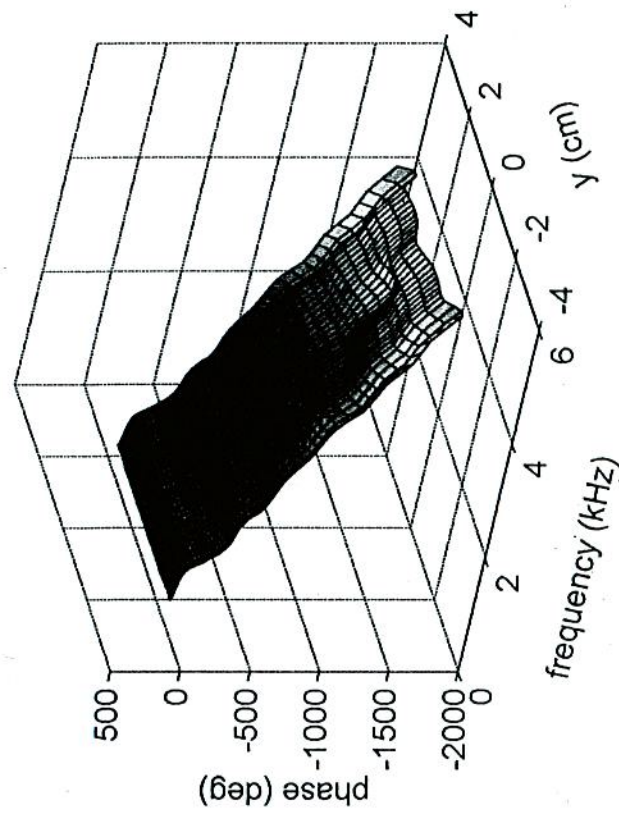
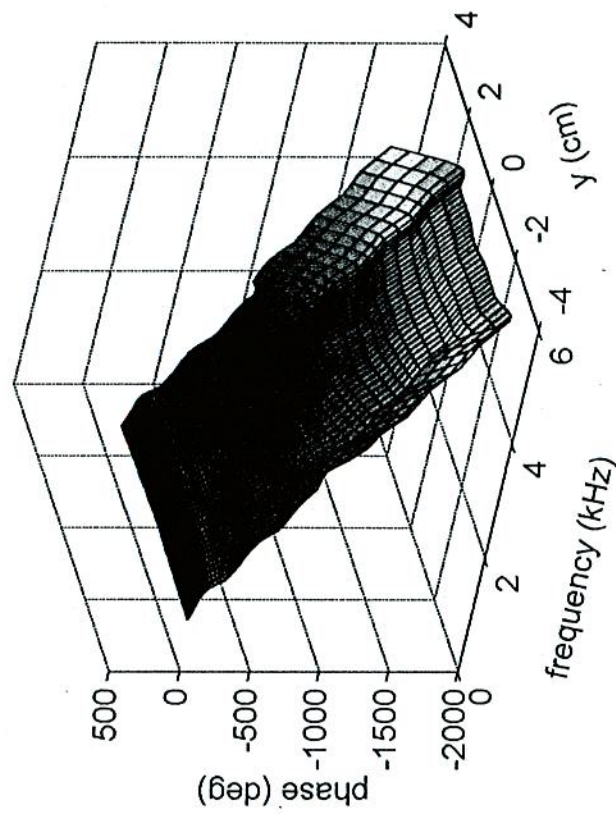
CONSTRAINED FLAT FOAM LAYER



CONSTRAINED FLAT FOAM LAYER



HAVING SPATIALLY GRADED TORTUOSITY



## CONCLUSIONS

- It has been found that the transmission loss of the wedge is significantly higher than that of a plane foam layer of the same volume in some high frequency bands.
- The TL appears to be enhanced by "converting" the incident plane wave into a non-radiating higher-order symmetric mode.
- The same increase in TL can be produced using a plane, constant depth foam layer if tortuosity is varied across the width of the foam layer.

