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REQUIREMENTS FOR A GOOD ACOUSTIC LABORATORY

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INTRODUCTION

It is a well-known fact that the compressor manufacturer in the course of time has been obliged to show some interest in the noise caused by his products.

The necessity becomes greater as the competition increases. It is becoming clear to the customer that noise is not absolutely necessary to achieve the primary object, namely refrigeration.

At the same time the authorities are setting up rules stating how much noise we may produce in and outside our houses. For refrigeration compressors it is fortunately a question of annoyance, rather than a question of high noise levels.

In the past a lot has been written about noise testing and noise evaluation methods, some usable, but also some so sophisticated that they were impossible to use in practice. In the last ten years a crystallization of the best of the methods has taken place, and within ISO usable standards and drafts are on their way (1), which is of great satisfaction. It would be gratifying if all those national standards could be replaced by international standards, thereby making it possible to speak the same language.

REQUIREMENTS

What is the requirements to a good acoustic laboratory from the point of view of a compressor manufacturer and his customer?

The primary object must be to own "a tool" which facilitates the manufacturing of a silent compressor.

1. For that purpose we shall of course need people with sufficient education and imagination to handle the problems in the correct manner.
2. Furthermore we shall have to purchase special measuring equipment in order to carry out the necessary detail analysis of the various components and parameters influencing the noise level. Things like valve examination, muffler calculation and investigation, spring and discharge tube analysis, and compressor shell investigation

tion can be mentioned.

3. We need measuring surroundings (test rooms) enabling us to carry out objective measuring with possibilities for sound power determination, with an accuracy as stated in international standards. The measuring room should also be acoustical adjusted (frequency independent reverberation time) so that it enables us to carry out a subjective evaluation of the noise character (pending, discrete frequencies, time variation, etc). Such an "adjustment" will also facilitate tape recordings of diverse equipment during testing and replay in front of an audience for pronouncement.

4. The measuring procedure must be practicable in time as it is normally necessary to carry out measurements on several samples, and at more than one running condition.

This is tantamount to measuring equipment with automatic data acquisition, which together with a digital computer admits an easy access for all desired calculating such as sound power level, noise evaluation, statistical tests, etc.

The following contains a review of the procedure with which we attempted to meet the requirements under points 3 and 4.

TYPES OF TEST ROOMS

It is commonly known that a reverberant room offers great advantages in regard to cost and test effort, compared with an anechoic room (2,3). The disadvantage is by and large limited to the problem of obtaining sufficient space averaging of the sound pressure when testing equipment whose spectra contains narrow bands of noise (4,5). Our reverberant test rooms (two identical rooms, constructed in 1968) are of the conventional rectangular type with data in accordance with several standards and draft proposals (6,7). The room volume is 200m³. Further data for the rooms are stated at the top right corner of figure 5. In order to be able to measure equipment with extremely low noise levels the rooms

are built with double walls. The inside room is vibration insulated by means of rubber springs. Background noise levels (room + measuring instruments) are stated as function of frequency in figure 5 (in table).

Reverberation time adjustment and diffusing elements.

The room were constructed with inside surfaces as hard as possible to create a high reverberation time at the high frequencies. After the finishing of the building, regulation of the room absorption was made for two reasons:

1. To increase the bandwidth of the resonance curves of the normal modes of the room, particularly at the low frequencies.
2. By and large to obtain a frequency-independent reverberation time for the possibility of making an auditive evaluation of noise from the test object as mentioned (see figure 1).

Most of the absorbing elements are made of perforated and unperforated aluminium panels. The elements were placed at random and at various angles to the room surface to work as diffusers, also. A final regulation was carried out by means of "tuned membrane absorbers" suspended in the room. Further diffusion was made in the ceiling construction (see figure 2).

There are now revolvable or oscillating diffusers in the room. Space averaging is made by a traversing microphone which moves with a constant velocity (0.4m/sec) over a path of two metres.

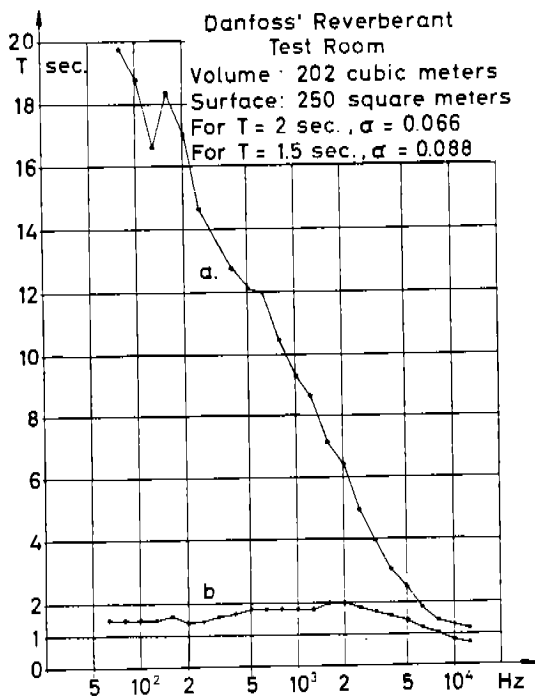


Fig.1 Reverberation Time versus Frequency
 a. Before regulation.
 b. After regulation.

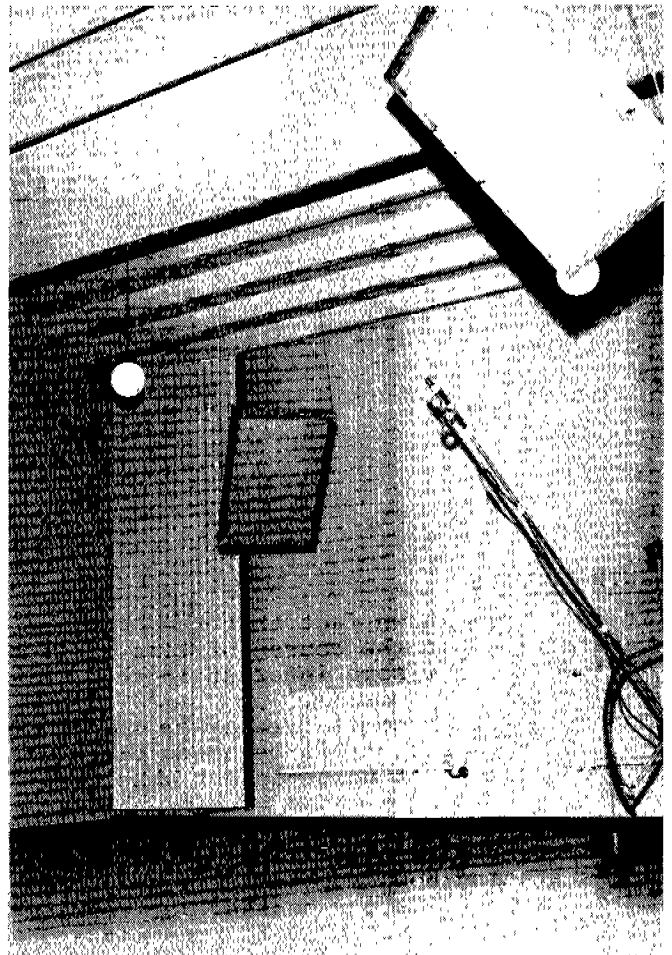


Fig. 2. Interior of test room.

INSTRUMENTATION

As measuring equipment are used a 1/3 octave band Real Time Analyser (B & K, type 3347) with continuous time averaging (RC-smoothing) with time constant equal to 20 seconds. The analyser is connected to a tape punch B&K type 5582). A Manual Data Extension Unit (B & K, type 5599) gives the possibility for information on sample type and number, running conditions, etc. (see figure 3). The data measured can then be calculated in a digital computer.

QUALIFICATION OF THE TEST FACILITIES

To examine the quality of the room, various measurements have been carried out on two different types of noise sources.

1. The conventionally used ILG-Reference Sound Source (broad band noise)
2. Selected compressor with narrow bands of noise and discrete frequencies.

Re.1. Figure 4 shows the sound pressure levels for the sound source (RSS) versus distance to microphone. From 125 Hz and up to 8 kHz we have very smooth sound pressure levels over a long distance except at 250 Hz. Qualification test for broad band noise

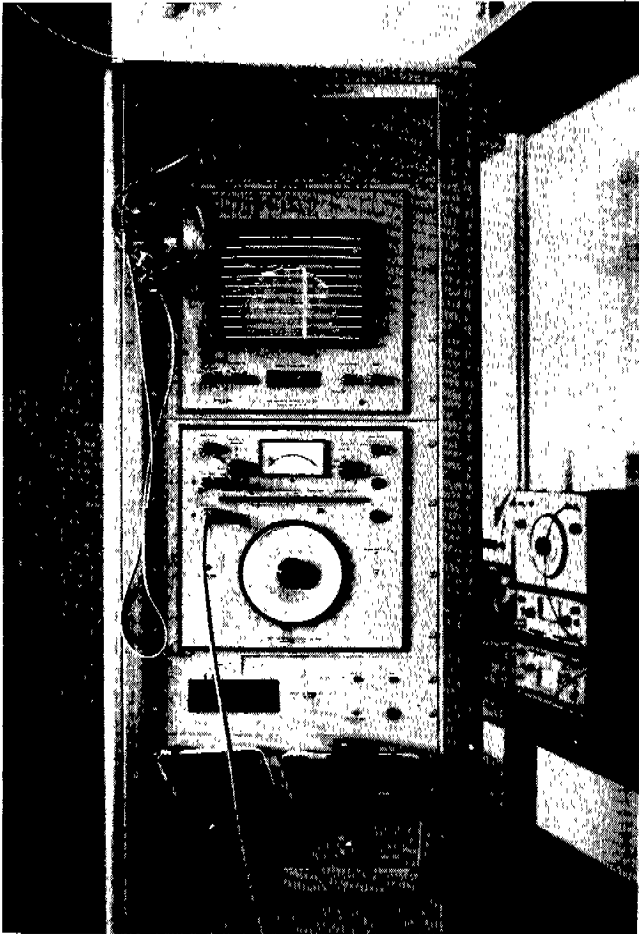


Fig. 3 Real-Time Analyser with Paper Tape Punch Unit.

measurements in accordance with the new ISO draft (7) are shown in figure 5a and 5b. The requirements are more than satisfied except the 100 Hz 1/3 octave band.

Other measurements (not shown here) show that one source location and the microphone traversed with constant velocity over a path of two metres, gives measuring uncertainty which is less than that mentioned in the new ISO-draft (7).

With a carefully selected fixed microphone location, the uncertainty will only be slightly increased, which is important when recording on tape recorders.

Re 2. When measuring sources with narrow bands of noise and discrete frequency, one must be more careful. Figure 6 shows measurements on a compressor, which, after the definitions in the ISO (7) part II, have narrow bands of noise with 315, 400, 800, 1600 and 2000 Hz and discrete frequencies with 500 and 1000 Hz.

Figure 7 shows measurements and statistical test with the compressor location moved 0.7 metres from the above mentioned. There were only significant deviations in a very few 1/3 octave bands. Finally, figure 8 shows a statistical test between the measurements from figure 6 and ten

measurements taken at intervals of one minute on the same compressor, but with fixed microphone located in the centre of the microphone path.

As expected there are significant differences between several frequency bands, but the deviations are modest.

PRESENTATION OF MEASURED AND CALCULATED DATA

Figure 5 to 8 also shows clear examples of how to present the measured sound pressure levels, calculated sound power levels and statistical test.

The applied equation for sound power level calculation

$$L_w = L_p + [10 \log_{10} V - 10 \log_{10} T - 14] \text{ dB}$$

has to be changed to

$$L_w = L_p + [10 \log_{10} V - 10 \log_{10} T +$$

$$10 \log_{10} (1 + \frac{S \cdot \lambda}{8 \cdot V}) - 14] \text{ dB}$$

to account for the effect of the interference pattern formed near the room surfaces, ISO part II (7).

FINALLY COMMENTS

Mentioned here are some of the most important requirements for a good acoustic laboratory, without, however, all the details.

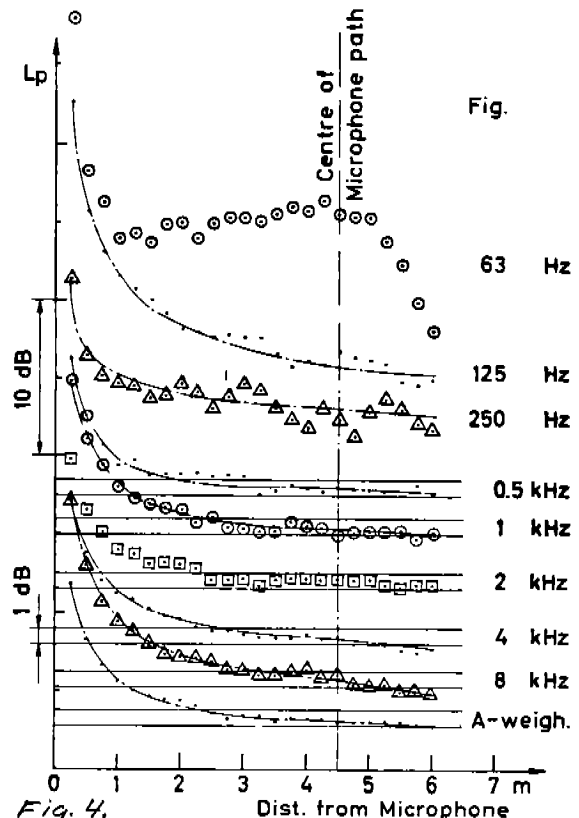


Fig. 4. Sound Pressure Levels for Reference Sound Source versus Distance to Microphone.

Of the factors which are significant for the sound power calculations are things like accuracy when adjusting the measuring equipment and the determination of reverberation time.

Oscillating or rotating diffusors are often used in reverberant test rooms, but one of the factors connected with this must be made clear. Such diffusors cannot operate during tape recordings, since space variations of the sound pressure level in the test room will be conceived as time variation of the test object.

The length of the microphone path we use is much less than demanded by ISO for measuring at discrete frequencies, but it is a question of whether this point is still too sophisticated?

It must be better having the other facilities in order, with the possibility to make compressors without pure tones than being able to measure with the utmost precision on a bad product!

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*** TEKNISK EDB, DANFOSS, NORDBORG. PROGRAM M3054 ***

 * NOISE MEASUREMENTS ON REFERENCE SOUND SOURCE *
 * WITH CALCULATION OF *
 * LP = MEAN SOUND PRESSURE LEVEL IN 1/3 OCTAVE BANDS *
 * S = STANDARD DEVIATION *
 * LW = MEAN SOUND POWER LEVEL IN 1/3- AND 1/1 OCTAVE BANDS *
 * CAL. AS LW = LP + (10 LOG V - 10 LOG T - 14) = LP + K (K SEE TABLE) *
 * LW(A) = THE WEIGHTED SOUND POWER LEVEL IN DB(A) *

TEST OBJECT	ILG.	RSS	* 220 VOLT *	ROOM	TEMP.	20 C *	MF. DATE	* CODE 1																		
11 MIK.POS.	1RSS.POS.	*	50 HZ	* 60% REL. HUM.		* MES.	-*- 03.06.72 *																			
T SEC	1.5	1.5	1.5	1.5	1.6	1.4	1.4	1.6	1.7	1.8	1.8	1.8	1.8	1.8	1.8	2.0	2.0	1.8	1.7	1.6	1.4	1.2	1.0	0.8	0.7	
LPBAC	19	15	12	10	12	10	8	8	7	6	6	5	5	5	5	5	5	6	6	6	6	6	6	6	6	
BAND	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	47
NUMBER	620	684	584	606	598	616	614	618	646	636	634	634	640	644	648	652	646	624	612	588	576	566	534	482	420	746
1000	620	690	568	594	590	614	612	618	640	634	634	636	642	644	650	658	646	622	612	586	576	562	532	480	414	746
900	602	666	562	594	586	618	606	620	640	642	642	642	643	642	646	646	660	644	624	610	586	572	562	534	484	746
800	610	682	552	582	588	618	604	612	640	630	648	632	642	642	644	646	646	642	620	610	588	574	566	536	484	742
700	606	668	550	580	582	612	616	616	644	630	648	642	644	646	644	644	650	648	622	610	590	576	558	534	462	746
600	600	664	544	570	584	608	620	612	628	632	640	642	640	636	644	652	646	622	610	586	572	560	532	478	404	744
500	592	660	544	560	582	612	624	624	632	636	638	638	642	638	642	652	644	624	608	584	572	556	530	476	402	742
400	586	652	538	556	576	606	606	620	628	636	634	634	646	638	646	648	642	618	604	582	568	550	528	472	400	740
300	582	638	534	548	576	618	614	636	634	646	644	634	640	638	644	644	644	620	608	586	570	550	530	472	400	742
200	580	632	532	540	570	612	606	632	626	634	640	636	642	640	642	640	640	618	606	582	570	554	526	470	396	740
100	580	616	526	540	570	618	616	632	626	634	630	642	646	638	644	644	642	618	608	584	570	556	528	470	396	742
LP#10	598	659	549	570	582	614	613	622	635	635	639	637	642	641	645	650	644	621	609	586	572	558	531	477	407	74.3
S	1.5	2.3	1.7	2.3	0.9	0.4	0.6	0.8	0.7	0.5	0.6	0.4	0.2	0.4	0.2	0.6	0.2	0.2	0.2	0.3	0.3	0.6	0.3	0.6	0.9	0.2
K	7.2	7.2	7.2	7.2	7.1	7.0	7.4	7.4	7.0	6.7	6.5	6.2	6.2	6.2	6.2	6.0	5.9	6.2	6.6	7.0	7.6	8.2	9.0	9.7	10.6	
LW1/3	670	731	621	642	653	684	687	696	705	702	704	699	704	703	707	710	703	683	675	656	648	640	621	574	513	
LW1/1	74.3			71.1			74.4				74.9				75.2			74.7		70.8			66.7			

THE WEIGHTED SOUND POWER LEVEL LW(A) = 80.5 DB(A)

Fig. 5 a.

*** TEKNISK EDB, DANFOSS, NORDBORG. PROGRAM M3054 ***

MEAN SOUND PRESSURE LEVEL, STANDARD DEVIATION AND SPECTROGRAM FOR
TEST OBJECT: 1LG. RSS * 220 VOLT * ROOM TEMP. 20 C * MF. DATE * CODE 1
11 MIK.POS. 1RSS.POS * 50 HZ * 60% REL. HUM. * MES. -- 03.06.72 *

HZ50 63 80 100 125 160 200 250 315 400 500 630 800 1K 1.2 1.6 2K 2.5 3.2 4K 5K 6.3 8K 10K 12K DBA
LP*10 598 659 549 570 582 614 613 622 635 635 639 637 642 641 645 650 644 621 609 586 572 558 531 477 407 74.3
S 1.5 2.3 1.7 2.2 0.8 0.4 0.6 0.8 0.7 0.4 0.5 0.3 0.2 0.3 0.2 0.6 0.2 0.2 0.2 0.2 0.5 0.3 0.5 0.8 0.2

SPECTROGRAM FOR CALCULATED MEAN SOUND PRESSURE LEVEL AND + 2 TIMES THE STD DEVIATION

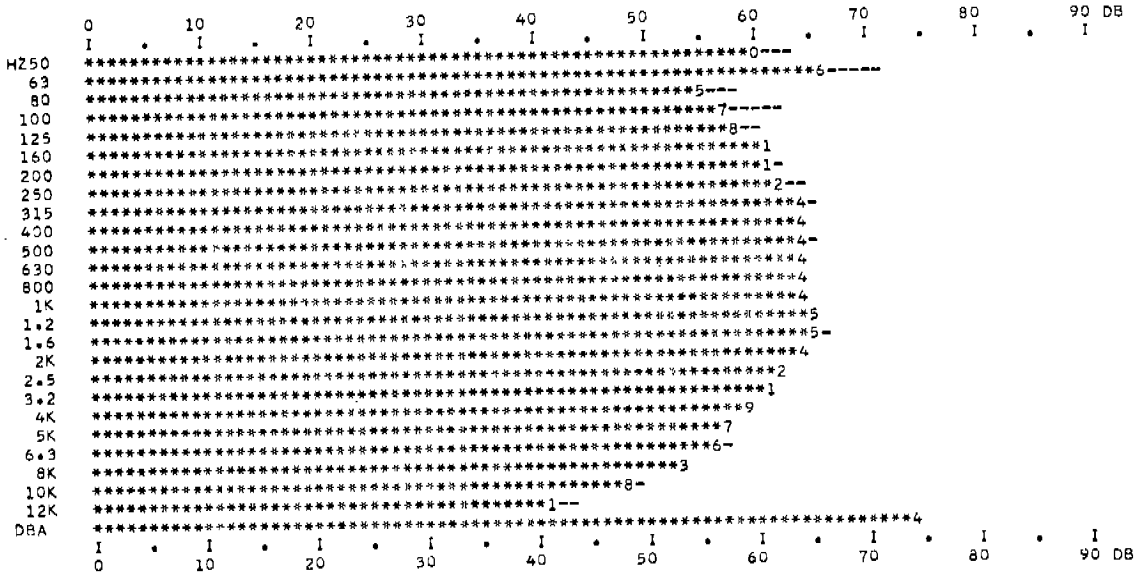


Fig. 5b.

*** TEKNISK EDB, DANFOSS, NORDBORG. PROGRAM M3054 ***

MEAN SOUND PRESSURE LEVEL, STANDARD DEVIATION AND SPECTROGRAM FOR
1 COMPRESSOR TYPE PW4.5K9 * 220 VOLT * EVAPORATING TEMP. -25 C * MF. DATE * CODE 6
1 SC.POS. 11 MIK.POS * 50 HZ * CONDENSING -- 55 C * MES. -- 03.06.72 *

HZ50 63 80 100 125 160 200 250 315 400 500 630 800 1K 1.2 1.6 2K 2.5 3.2 4K 5K 6.3 8K 10K 12K DBA
LP*10 193 145 133 121 120 130 125 137 148 157 251 122 121 158 113 178 273 204 169 137 120 106 110 98 111 31.6
S 1.0 0.2 0.2 0.3 0.7 0.4 1.2 0.7 1.8 2.3 3.7 1.1 1.7 3.2 0.5 1.7 1.8 0.7 1.2 0.5 0.2 0.2 0.0 0.1 0.2 1.3

SPECTROGRAM FOR CALCULATED MEAN SOUND PRESSURE LEVEL AND + 2 TIMES THE STD DEVIATION

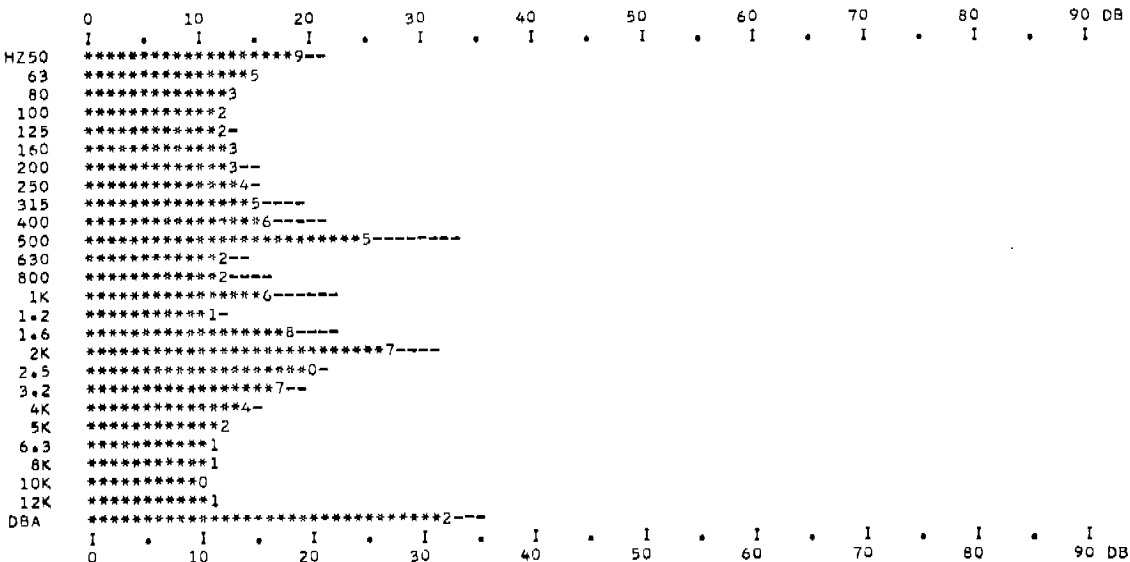


Fig. 6

