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# Modelling high velocity impact on thin woven composite plates: A non-dimensional theoretical approach

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## Abstract

A new theoretical energy-based model that predicts the ballistic behaviour of thin woven composite laminates is presented. This model formulated for high-velocity impacts, where the boundary conditions (applied at the external edges of the impacted plate) do not play a relevant role. This can be assumed as the mechanical waves do not reach the borders during the impact event, being the local structural behaviour the responsible for the ballistic performance. A non-dimensional formulation is used to analyse the influence of material properties and geometrical parameters in the ballistic response of the laminate. The model is physically-based on the energy contribution of different energy-absorption mechanisms. A 3D finite element model previously developed is used to simulate the performance of the laminate under high-velocity impacts and to validate the hypotheses of the theoretical model. A comparison between FE and theoretical models is performed by means of energy absorption mechanisms. For that, the failure modes of the FE model are related to the corresponding energy-absorption mechanisms of the theoretical associated. The evaluation of the theoretical results is straightforward although the FEM results require the evaluation of the energy absorbed by each element that fails under each criterion. The predictive capability of the proposed model is verified against experimental results, which were obtained from previous studies carried out by the authors. The results obtained show the dependencies between the ballistic response and the non-dimensional physical parameters of the model. Furthermore, the proposed model can be used to see the relative importance of the different energy-absorption mechanisms involved and the comparison of these mechanisms between the theoretical and the FE models can reflect the different roles played by them, depending on the material properties and geometrical characteristics of the laminate. These results highlight the relevance of the in-plane energy-absorption mechanisms, which rule the penetration process for thin laminates.

*Keywords:* Woven composites, Ballistic response, Theoretical modelling, Numerical modelling, Energy-based analysis

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## 1. Introduction

Composite materials are suitable for a large number of lightweight structural applications, namely in transports engineering, where weight reductions, and consequently fuel efficiency, are key priorities (Llorca et al., 2011). Consequently, it is essential to pay attention to the response of these materials under extraordinary circumstances such as impacts that can affect their mechanical properties and compromise their performance. In-plane mechanical properties of composite laminates, however, are known to quickly deteriorate when subjected to out-of-plane impact loads (Alonso et al., 2018b). There are different approaches to tackle the impact phenomenon: experimental, numerical and analytical. Although experimental and finite element approaches are still the preferred methods to study the impact behaviour of composite materials,

43 their use is often limited by high computational costs (Miami et al., 2007). On the other hand, analytical  
44 and theoretical approaches, as shown in this paper, are good alternatives to understand the physics of the  
45 problem and provide sufficiently accurate solutions at lower computational cost than numerical methods  
46 (Naik and Doshi, 2005).

47 Most analytical approaches predict the residual velocity through conservation of momentum and energy  
48 balances. Models exclusively based on momentum transfer are physically consistent but are limited in  
49 providing information on specific energy-absorption mechanisms (Briescani et al., 2015; Mamivand and  
50 Liaghat, 2010). Models that include energy balances, however, provide fundamental insights on the different  
51 failure mechanisms. This has motivated a formulation based on energy-absorption mechanisms in this  
52 study. The level of detail is proportional to the complexity of the mechanisms described by the analytical  
53 equations and the initial hypotheses considered. Regarding energy-based theoretical models, the current  
54 literature is divided between models for thin and thick laminates. Often, the initial hypotheses considered  
55 (behaviour of the laminate, energy-absorption mechanisms considered and neglected) are different depending  
56 on the thickness, leading to diverse formulations. In a previous study carried out by the authors (Alonso  
57 et al., 2018a), the threshold from which a laminate changes its structural response from thin to thick  
58 was found at laminate thickness close to the projectile diameter. Analytical models that can predict the  
59 ballistic performance of thin laminates often consider non-spherical projectile geometries and are limited  
60 to in-plane failure modes: elastic deformation of fibres, tensile failure of fibres, movement of the laminate,  
61 matrix cracking and delamination (Zhu et al., 1992; Navarro, 1998; Moyre et al., 2000; Naik and Shirao,  
62 2004; García-Castillo et al., 2013). These models have been further extended to predict the response of  
63 thick composite plates, incorporating additional energy absorption mechanisms such as shear plugging and  
64 crushing while neglecting others (Naik and Shirao, 2004; Naik and Doshi, 2005; Naik et al., 2005, 2006). A  
65 number of different approaches can be found in the literature that also consider the shape of the projectile,  
66 observing significant differences in the energy absorption of thick laminates impacted by blunt, conical and  
67 truncated projectiles (Wen, 2000, 2001). Nevertheless, studies considering diverse shapes of the projectile  
68 when impacting thin laminates do not show remarkable differences in the ballistic response. Regarding thin  
69 laminates, further research is required to capture the penetration rate and the relative contribution of each  
70 mechanism in the final energy absorption capacity of the laminate.

71 One of the issues hindering the implementation of more accurate analytical models is the difficulty to  
72 introduce new hypotheses, such as the relative displacement between the projectile and the laminate to define  
73 the perforation process, which is newly introduced in this paper, to the best of our knowledge, for high-  
74 velocity impact models. High-speed cameras do not provide sufficient resolution to ascertain the physical  
75 processes in the first instants of the impact event and, therefore, can not be used to validate the contribution  
76 of different failure mechanisms (Gellert et al., 2000; Buitrago-Pérez et al., 2010). In this context, numerical  
77 modelling is a complementary tool to clarify the micro-mechanisms involved in complex problems such as  
78 impact (Moyre et al., 2000; Naik and Doshi, 2005; López-Puente et al., 2007; Briescani et al., 2015; Alonso  
79 et al., 2018a). Some authors have combined analytical and numerical models, also using the numerical  
80 model as a tool to validate the ballistic results of the analytical. (Mohotti et al., 2015; Gregori et al.,  
81 2020) developed analytical models for aluminium–polyurea composite layered plates and ceramic-composite  
82 targets respectively and validated them with finite element simulations with the software LS-DYNA.

83 This work aims at characterising the ballistic response of thin GFRP laminates through theoretical  
84 modelling. To this end, a non-dimensional theoretical model is proposed, based on physical energy-based  
85 hypotheses. The model provides information about the ballistic behaviour and the energy-absorption mech-  
86 anisms. The impact vs residual velocity curve is compared with experimental data from the literature  
87 (Buitrago-Pérez et al., 2010). The present model incorporates, for the first time, new hypotheses which  
88 allow for the prediction of the penetration rate and final indentation in the laminate. Although the analysis  
89 focuses on spherical projectiles, the model is formulated so that it accounts for different projectile geometries.  
90 Furthermore, a numerical constitutive model for woven composites, previously developed by the authors  
91 (Alonso et al., 2020), is used to validate the theoretical assumptions behind the proposed formulation. The  
92 model proposes a continuum damage mechanics approach based on a maximum stress criteria. The energy  
93 absorbed by the elements that fail under each criterion is related to the corresponding energy-absorption  
94 mechanism or a combination of some in the theoretical model. The failure criteria in the FEM as well as

the energy-absorption mechanisms in the theoretical model can be classified into in-plane and out-of-plane failure criteria/energy-absorption mechanisms. The proposed analytical model is an efficient tool to evaluate both the ballistic response and transitions in the relative roles played by specific energy-absorption mechanisms, depending on the material properties and geometrical characteristics of the plate. These outcomes are useful to address the first design and optimisation stages of structural components subjected to impact loading.

## 2. Theoretical model

The energy-based theoretical model developed to predict the ballistic performance of thin woven E-glass/polymer laminates is presented in this section. The model assumes that the kinetic energy of the projectile is partly dissipated during the perforation process and partly transferred to the composite laminate. The model hypotheses, the kinematics of the perforation process and the formulation of the energy absorption mechanisms are described in the following paragraphs.

This model is based on a previous non-dimensional energy-based model that takes into account the same energy-absorption mechanisms but assumes simpler hypotheses (Alonso et al., 2018a). To identify the model parameters, a non-dimensional formulation is used based on the Vaschy-Buckingham II theorem. The model depends on three elemental magnitudes: the mass [M], length [L] and time [T], which can be written in non-dimensional form as

$$[M] = \rho_p \phi_p^3 \quad (1)$$

$$[L] = \phi_p \quad (2)$$

$$[T] = \frac{\phi_p}{V_i} \quad (3)$$

where  $\rho_p$  is the density of the projectile material,  $\phi_p$  is the projectile diameter and  $V_i$  is the impact velocity.

The following hypotheses are proposed. Some of them are based on previous experimental observations (Buitrago-Pérez et al. (2010); García-Castillo et al. (2006); Alonso et al. (2018a)):

- The projectile is rigid and, as such, remains undeformed during the impact.
- The laminate has linear-elastic behaviour and is  $x$ -axially symmetric ( $x$  is the thickness direction).
- Wave speeds do not change during the impact.
- The laminate is accelerated by the projectile.
- During penetration, the laminate moves with a different velocity from that of the projectile. Consequently, there is a relative displacement between the projectile and the laminate.
- The model accounts for the time before the relative displacement between the projectile and the laminate equals the thickness of the laminate. From this moment onward, no further energy transfer is considered.
- The energy dissipated through tensile failure and elastic deformation of the fibres is accounted for separately.
- The energy dissipated by friction, shear plugging and heat transfer is negligible.

As mentioned before, the model is formulated in a non-dimensional way. This formulation leads to the apparition of the parameters defined in Table 1. The dynamic properties at high strain rates, presented in Table 1, used in the theoretical and numerical models are estimated from the static properties obtained in Alonso et al. (2018a, 2020) taking into account the relations proposed by Harding and Welsh (1983); Harding and Ruiz (1998). The high-strain rate correction factors for the failure limits and for the shear and Young's moduli are estimated at 1.5 and 3, respectively. .

Table 1: Summary of the parameters and  $\Pi$  groups of the problem.

Parameter	Nomenclature	Value	$\Pi$ group
Projectile diameter	$\phi_p$	7.5 [mm]	
Projectile density	$\rho_p$	7809 [kg.m <sup>-3</sup> ]	
Impact velocity	$v_i$ [m.s]		
Laminate thickness	$e$		$\Pi_e = \frac{e}{\phi_p}$
In-plane Young's modulus	$E$	15.2 [GPa]	$\Pi_E = \frac{E}{\rho_p v_i^2}$
In-plane failure strain	$\varepsilon_r$	0.0725	$\varepsilon_r$
In-plane failure stress	$\sigma_r$	1.102 [GPa]	$\Pi_{\sigma_r} = \frac{\sigma_r}{\rho_p v_i^2}$
Laminate density	$\rho_l$	1980 [kg.m <sup>-3</sup> ]	$\Pi_{\rho_l} = \frac{\rho_l}{\rho_p}$
Absorbed energy density by matrix cracking	$E_{MT}$	10 <sup>6</sup> [J.m <sup>-3</sup> ]	$\Pi_{E_{MT}} = \frac{E_{MT}}{\rho_p v_i^2}$
Critical dynamic-strain energy-release rate in mode II	$G_{IICD}$	3000 [J.m <sup>-2</sup> ]	$\Pi_{G_{IICD}} = \frac{G_{IICD}}{\rho_p \phi_p v_i^2}$
Yarn width	$B$	5 [mm]	$\Pi_B = \frac{B}{\phi_p}$
Stress wave transmission factor	$b$	0.9	$b$
Poisson's ratio	$\nu$	0.16	$\Pi_\nu = [12(1 - \nu^2)]^{1/6}$
Shape factor of delamination	$\alpha_{DL}$	1	$\alpha_{DL}$
Shape factor of matrix cracking	$\alpha_{MC}$	1	$\alpha_{MC}$
Constant	$c$	0.25 [N.m] <sup>-1/6</sup>	$\Pi_c = c \phi_p^{1/2} \rho_p^{1/6} V_i^{1/3}$

135 As shown in Table 1, the parameters combinations which govern the problem can be associated with  
136 physical meaning. These combinations are known as the  $\Pi$  groups of the problem. In this model, the  
137 mechanics of the impact and penetration depends on 16 fundamental parameters which are listed in Table 1,  
138 together with their corresponding  $\Pi$  groups associated with them. In view of Table 1, note that some of the  
139 parameters ( $\varepsilon_r$ ,  $b$ ) are inherently non-dimensional, so the parameter is directly the  $\Pi$  group in those cases.  
140 Eventually, after applying the Vaschy-Buckingham Theorem, the problem depends on 13  $\Pi$  groups.

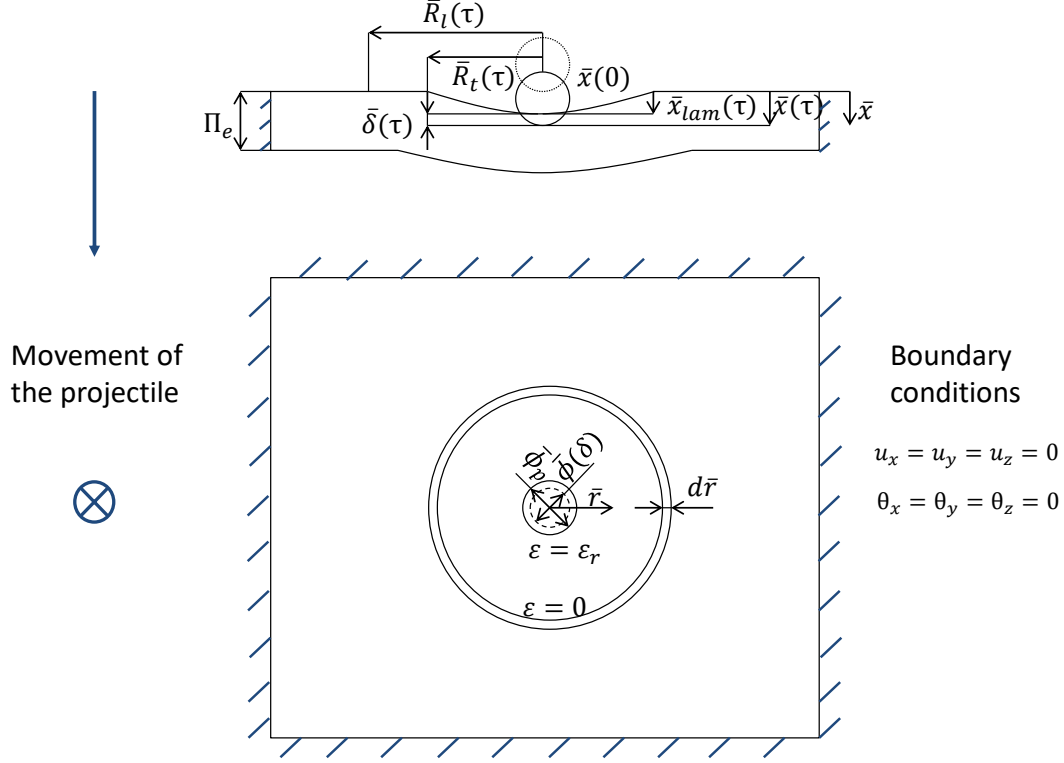


Figure 1: Schematic representation of a generic time instant of the impact process, showing the relevant variables.

141 Figure 1 shows a schematic representation of a generic instant of the impact process, where the variables  
 142 and parameters described in the following sections can be easily identified. Note that an overbar designates  
 143 a non-dimensional variable. The non-dimensional kinematic variables of the problem are: the projectile  
 144 position  $\bar{x}(\tau)$ , the projectile velocity  $\bar{v}(\tau)$  and the projectile acceleration  $\bar{a}(\tau)$ . The non-dimensional time  $\tau$   
 145 is the integration variable.

### 146 2.1. Wave propagation and model

147 From the one dimensional wave theory (Smith et al., 1958), it can be stated that when a fibre is trans-  
 148 versely impacted two waves are generated and propagate: (i) a longitudinal wave,  $C_l$ , which induces a steady  
 149 tensile strain and travels at the elastic wave speed of the material, and (ii) a transverse wave,  $C_t$ , responsible  
 150 for the acceleration of the laminate. These waves can be described as

$$C_l = \sqrt{\frac{\Pi_E}{\Pi_{\rho_l}}} \quad (4)$$

$$C_t = \sqrt{(1 + \varepsilon_r) \frac{\Pi_{\sigma_r}}{\Pi_{\rho_l}}} - \sqrt{\frac{\Pi_E}{\Pi_{\rho_l}}} \varepsilon_r \quad (5)$$

152 where  $\Pi_E$ ,  $\Pi_{\sigma_r}$ ,  $\varepsilon_r$  and  $\Pi_{\rho_l}$  are the  $\Pi$  groups related to the Young's modulus, the in-plane failure stress,  
 153 the in-plane failure strain and the laminate density, respectively. The transverse wave speed equation 5 is

154 obtained as function of the steady strain for each particular impact velocity, considering fibres are linear-  
 155 elastic before failure Smith et al. (1958). Inelastic waves may be also generated when a fibre is impacted.  
 156 However, it seems a reasonable simplification to limit the analysis to the elastic waves, motivated on the  
 157 almost perfectly linear-elastic response until failure of the in-plane mechanical behaviour of the laminate.  
 158

In Figure 1, the radii of the regions under longitudinal and transverse waves are, respectively,

$$\bar{R}_l(\tau) = C_l \tau \quad (6)$$

$$\bar{R}_t(\tau) = C_t \tau \quad (7)$$

159  
 160 Figure 2 shows the particle velocities at different locations on the laminate. Two different regions can  
 161 be distinguished. The first one is dominated by the longitudinal wave, bounded between the longitudinal  
 162 and transverse wave fronts (from point 2 to 3), while second region is dominated by the transverse wave,  
 163 delimited by the transverse wave front (from point 1 to 2). Conventional models consider the longitudinal  
 164 and transverse particle velocities of an impacted plate such as Moyre et al. (2000), Naik and Shrirao (2004),  
 165 García-Castillo et al. (2009). The proposed model considers instead that the radial particle velocity,  $\bar{v}_r(\tau)$ , is  
 166 constant between points 2 and 3, and decreases linearly to zero at the impact location (point 1). The particle  
 167 transverse velocity,  $\bar{v}_l(\tau)$ , is assumed to be maximum at the impact (point 1), decreasing until reaching the  
 168 transverse wave front, as shown in Figure 2.

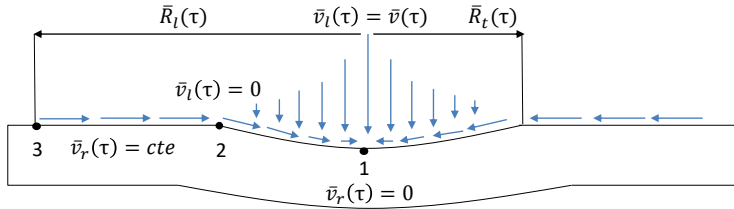


Figure 2: Schematic representation of the transverse and radial particle motion in the laminate.

169 With the progress of penetration through the laminate, the distance between the mid-plane of the plate  
 170 and the centre of mass of the projectile decreases. The fibres fail through the thickness with the relative  
 171 penetration of the projectile, neglecting potential out-of-plane mechanisms following the assumption of  
 172 membrane behaviour. The penetration,  $\bar{\delta}$ , follows the relationship

$$\bar{\delta}(\tau, \bar{x}(\tau)) = \bar{x}(\tau) - \int_0^\tau \bar{v}_l(\tau) d\tau \quad (8)$$

173 where  $\bar{x}(\tau)$  is the location of the centre of mass of the projectile, and  $\bar{v}_l(\tau)$  is an average velocity of the plate  
 174 measured at the mid-plane. This velocity can be estimated by different approaches. When a high-velocity  
 175 impact takes place and the transverse wave propagates, the movement of the laminate as a membrane is  
 176 governed by a profile of velocities. In the contact point between the two bodies the velocity is the one  
 177 of the projectile and a gradient is assumed up to the point reached by the transverse wave, where the  
 178 transverse velocity is zero. Nevertheless, the accurate measurement of this profile with the experimental  
 179 devices available nowadays is an impossible task. That is why we propose here a phenomenological function  
 180  $\bar{k}(\tau, \bar{v}(\tau)) \in (0, 1]$ , depending on the non-dimensional groups of the problem, instead of a profile of velocities.  
 181 This function represents the percentage of the projectile velocity at which the laminate moves. Assuming this  
 182 simplification, a constant velocity of the laminate as a whole can be estimated to calculate the penetration  
 183  $\bar{\delta}$ . This hypotheses will be checked with the FE model. Aiming at understanding the physically motivated  
 184 ratios on which this phenomenological function depends, an explanation of the dimensional version of the  
 185 function  $k(t, v(t)) \in (0, 1]$  is given as

$$k(t, v(t)) = cD^{1/6} \left( \frac{v(t)}{v_i} \right)^2 \left( \frac{v_i t}{e} \right)^{1/2}, k \in (0, 1] \quad (9)$$

187 where  $D$  is the flexural rigidity of a plate defined in equation 10,  $c$  is a constant value of  $0.25 \text{ [N.m]}^{-1/6}$   
 188 and  $v_t$  is the transverse wave

$$D = \frac{Ee^3}{12(1-\nu^2)} \quad (10)$$

189 The function  $k(t, v(t))$ , can, therefore, be split into three parts. Attending to the flexural rigidity, the  
 190 higher  $D$ , the higher  $k(t, v(t))$ . Therefore, the membrane behaviour will be more difficult to appreciate  
 191 because the relative displacement will be very high and the laminate will have less time to bend. This  
 192 behaviour can be observed in laminates where the stiffness is higher and, instead of a cone, a plug is formed  
 193 (Cantwell and Morton, 1990; Kim et al., 2003; Rhymer et al., 2012). The relationship between  $v(t)$  and  $V_i$   
 194 seems to be reasonable because at the beginning of the impact, when there is no damage, this ratio is higher  
 195 leading to a  $k(t, v(t))$  increase and the laminate moves almost like the projectile. Moreover, when the fibres  
 196 and the matrix start to break, the relative displacement increases and also the difference between velocities,  
 197 thus lowering this ratio. Finally, the ratio between the radius of the transverse wave and the thickness has  
 198 a clear meaning because the lower this ratio (when the thickness increases), the lower the function  $k(t, v(t))$   
 199 and it happens when the thickness increases. Therefore,  $k(t, v(t))$  is lower, leading to higher difficulties  
 200 to appreciate the projectile and the laminate moving together (membrane behaviour) when the thickness  
 201 increases.  
 202

203 Rewriting  $k(t, v(t))$  as a function of the  $\Pi$  groups of the problems leads to equation 11

$$\bar{k}(\tau, \bar{v}(\tau)) = \frac{\Pi_c}{\Pi_\nu} \Pi_E^{1/6} C_{V_t}^{1/2} \bar{v}(\tau)^2 \tau^{1/2}, \bar{k} \in (0, 1] \quad (11)$$

204 Finally, indentation during impact leads to a change of the contact surface between the projectile and  
 205 the plate, resulting in an equivalent projected diameter

$$\bar{\phi}(\bar{\delta}) = \begin{cases} 2\sqrt{\bar{\delta} - \bar{\delta}^2}, & \text{if } \bar{\delta} < 0.5 \\ 1, & \text{if } \bar{\delta} \geq 0.5 \end{cases} \quad (12)$$

## 206 2.2. Energy absorption mechanisms

207 The general expression to calculate kinetic energy absorbed by the acceleration of the laminate,  $\bar{E}_L$ ,  
 208 assuming a certain profile of velocities can be described as

$$\bar{E}_L = \pi \Pi_e \Pi_{\rho_l} \int_0^{\bar{R}_t(\tau)} r \bar{v}(r, \tau)^2 d\bar{r} \quad (13)$$

209 where the integral can be simplified by the equivalent laminate velocity at the mid-plane as it was explained  
 210 in last section, leading to

$$\bar{E}_L(\tau) = \frac{1}{2} \pi \Pi_e \Pi_{\rho_l} \bar{R}_t(\tau)^2 \bar{v}_l(\tau)^2 \quad (14)$$

211 The second mechanism is the elastic energy absorbed by fibres,  $\bar{E}_F$ , corresponding to the area below the  
 212 in-plane stress-strain curve, which can be calculated as

$$\bar{E}_F = \frac{1}{2} \Pi_E \varepsilon^2 \quad (15)$$

213 The contribution to this energy mechanisms can be divided in two different groups: (i) directly impacted,  
 214  $\bar{E}_{TF}$ , and (ii) adjacent fibres,  $\bar{E}_{ED}$ , (Moyre et al., 2000). Directly impacted fibres undergo tensile deforma-  
 215 tion due to the propagation of the longitudinal wave. Considering that the fibres break as the projectile  
 216 goes through them, we can express a differential energy as a function of a differential relative displacement:

$$d\bar{E}_{TF} = \Pi_B d\bar{\delta} \int_0^{\bar{R}_l(\tau)} \frac{1}{2} \Pi_E \varepsilon^2 d\bar{r} \quad (16)$$



217 where  $\Pi_B$  is the  $\Pi$  group related to the yarn width. The strain gradient along the yarn direction can be  
 218 defined by the following expression, proposed by Naik et al. (2006):

$$\varepsilon = \varepsilon_r b^{\frac{\bar{r}}{\Pi_B}} \quad (17)$$

219 where  $\bar{r}$ ,  $b$  and  $\varepsilon_r$  are the radial coordinate, the stress wave transmission factor and the in-plane failure  
 220 strain, respectively. The strain is maximum at the impact point, and decays radially. If maximum strain is  
 221 at the impact point, failure breakage is initiated at the impact point.

222 The volume of fibres involved in tensile failure,  $\bar{V}_{TF}$ , is driven by the penetration of the projectile and  
 223 the circular area given by the radius of the longitudinal wave. The projectile is assumed to be big enough  
 224 to impact on two perpendicular yarns directly with an inherent volume:

$$\bar{V}_{TF}(\tau) = 4\Pi_B \bar{\delta}(\tau) \bar{R}_l(\tau) \quad (18)$$

225 Therefore, the final elastic energy absorbed by fibre failure can be obtained considering the symmetry of  
 226 the cross-ply laminate. Dividing equation 16 with respect to non-dimensional time and rearranging terms,  
 227 provides:

$$\bar{E}_{TF}(\tau) = \frac{\Pi_B \Pi_E \varepsilon_r^2}{\ln(b)} \int_0^\tau \bar{v}(\tau) \{1 - \bar{k}[\tau, \bar{v}(\tau)]\} \left( b^{\frac{2C_l \tau}{\Pi_B}} - 1 \right) d\tau \quad (19)$$

228 Adjacent fibres, bounded by the contact area of the projectile  $\bar{\phi}(\bar{\delta})$  and the transverse wave front, will be  
 229 under linear elastic deformation defined by a linear gradient, which assumes maximum strains are reached  
 230 at the periphery of the projectile and zero at the transverse wave front, that is,

$$\varepsilon = \varepsilon_r \left[ \frac{2(\bar{R}_t(\tau) - \bar{r})}{2\bar{R}_t(\tau) - \bar{\phi}(\bar{\delta})} \right] \quad (20)$$

231 with an associated volume,  $\bar{V}_{ED}$ , given by:

$$\bar{V}_{ED}(\tau) = \pi \Pi_e \left( \bar{R}_t(\tau)^2 - \frac{\bar{\phi}(\bar{\delta})^2}{4} \right) \quad (21)$$

232 resulting in the final expression:

$$\bar{E}_{ED}(\tau) = \pi \Pi_e \Pi_E \int_{\frac{\bar{\phi}(\bar{\delta})}{2}}^{\bar{R}_t(\tau)} \varepsilon^2 \bar{r} d\bar{r} \quad (22)$$

233 Additional energy absorption mechanisms have been identified as matrix cracking,  $\bar{E}_{MC}$ , and delami-  
 234 nation,  $\bar{E}_{DL}$ . Matrix failure is controlled by the transverse wave and bending of the laminate, resulting in  
 235 a circular shaped damaged area (Alonso et al., 2018b; Gil-Alba et al., 2019). Total energies absorbed by  
 236 matrix cracking and delamination are defined, respectively as

$$\bar{E}_{MC}(\tau) = \pi \alpha_{MC} \Pi_e \Pi_{EMT} \bar{R}_t(\tau)^2 \quad (23)$$

$$\bar{E}_{DL}(\tau) = \pi \alpha_{DL} \Pi_{GICD} \bar{R}_t(\tau)^2 \quad (24)$$

238

### 239 2.3. Energy balance

240 The model is formulated by means of an energy balance. This balance may be written in its dimensional  
 241 form as

$$E_0 = E_p(t) + E_{AB}(t) \quad (25)$$

242 This instantaneous balance is valid for any time  $t$  and implies that the initial kinetic energy of the pro-  
 243 jectile,  $E_0$ , is equal to the sum of the kinetic energy of the projectile at any time instant,  $E_p(t)$ , to the

244 energy absorbed by all the mechanisms described above at that same time,  $E_{AB}(t)$ . In the non-dimensional  
 245 formulation, this balance becomes

$$1 = \bar{v}(\tau)^2 + \frac{12}{\pi} \bar{E}_{AB}(\tau) \quad (26)$$

246 where  $\bar{E}_{AB}(\tau)$  is

$$\bar{E}_{AB}(\tau) = \bar{E}_L(\tau) + \bar{E}_{ED}(\tau) + \bar{E}_{TF}(\tau) + \bar{E}_{DL}(\tau) + \bar{E}_{MC}(\tau) \quad (27)$$

247 The balance of Eq. (27) is derived with respect to the non-dimensional time providing the expression of Eq.  
 248 (28) with its corresponding initial conditions:

$$\bar{a}(\tau) = \frac{\bar{g}(\tau, \bar{x}(\tau), \bar{v}(\tau)) - \bar{h}(\tau, \bar{v}(\tau))\bar{v}(\tau)}{\frac{\pi}{6}\bar{v}(\tau) + \pi\Pi_e\Pi_{\rho_l}C_{V_t}^2[2\tau^{5/2}\bar{k}(\tau, \bar{v}(\tau))\frac{\Pi_e}{2\Pi_\nu}\Pi_E^{1/6}C_{V_t}^{1/2}\bar{v}(\tau)^3 + \tau^2\bar{k}(\tau, \bar{v}(\tau))^2\bar{v}(\tau)]} - \frac{\pi\Pi_e\Pi_{\rho_l}C_{V_t}^2[\tau\bar{k}(\tau, \bar{v}(\tau))^2\bar{v}(\tau)^2 + \tau^{3/2}\bar{k}(\tau, \bar{v}(\tau))\frac{\Pi_e}{2\Pi_\nu}\Pi_E^{1/6}C_{V_t}^{1/2}\bar{v}(\tau)^4]}{\frac{\pi}{6}\bar{v}(\tau) + \pi\Pi_e\Pi_{\rho_l}C_{V_t}^2[2\tau^{5/2}\bar{k}(\tau, \bar{v}(\tau))\frac{\Pi_e}{2\Pi_\nu}\Pi_E^{1/6}C_{V_t}^{1/2}\bar{v}(\tau)^3 + \tau^2\bar{k}(\tau, \bar{v}(\tau))^2\bar{v}(\tau)]} \quad (28)$$

$$\begin{aligned} \bar{x}(0) &= 0 \\ \bar{v}(0) &= 1 \end{aligned}$$

249 The stop condition of the model is:

$$\bar{\delta}(\tau) = \Pi_e \quad (29)$$

250 Functions (30) and (31) are defined to facilitate the handling of equations:

$$\bar{h}(\tau, \bar{v}(\tau)) = \frac{d\bar{E}_{TF}(\tau)}{d\tau} \frac{1}{\bar{v}(\tau)} \quad (30)$$

$$\bar{g}(\tau, \bar{x}(\tau), \bar{v}(\tau)) = \begin{cases} -\frac{d}{d\tau}[\bar{E}_{DL}(\tau) + \bar{E}_{MC}(\tau)], & \text{if } \tau \leq \frac{\bar{\phi}(\bar{\delta})}{2C_{V_t}} \\ -\frac{d}{d\tau}[\bar{E}_{ED}(\tau) + \bar{E}_{DL}(\tau) + \bar{E}_{MC}(\tau)], & \text{if } \tau > \frac{\bar{\phi}(\bar{\delta})}{2C_{V_t}} \end{cases} \quad (31)$$

252 This second-order non-linear differential equation can be solved by numerical integration. Integrating this  
 253 equation, the velocity and the position of the projectile can be obtained. Furthermore, once the problem is  
 254 solved, by substituting the outputs in Eq. (28), the acceleration of the projectile is obtained and thus all  
 255 the kinematic variables of the problem.

### 256 3. Numerical modelling

257 To fully validate the theoretical model presented in the previous section, a 3D numerical model previously  
 258 developed by the authors has been additionally used (Alonso et al., 2020), [where all the properties needed for  
 259 this model can be found](#). This model is based on a continuum damage mechanics approach and defines the  
 260 failure criteria with equivalences to the ones contemplated by the theoretical model (Hashin, 1980; Muñoz  
 261 et al., 2015; Chang and Chang, 1987; Menna et al., 2011). The constitutive response of the material is  
 262 linear-elastic up to the onset of damage and can be described in Mandel's notation as

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{13} \end{bmatrix} = \begin{bmatrix} \frac{1}{E_{11}(1-d_1)} & -\frac{\nu_{21}}{E_{22}} & -\frac{\nu_{31}}{E_{33}} & 0 & 0 & 0 \\ -\frac{\nu_{12}}{E_{11}} & \frac{1}{E_{22}(1-d_2)} & -\frac{\nu_{32}}{E_{33}} & 0 & 0 & 0 \\ -\frac{\nu_{13}}{E_{11}} & -\frac{\nu_{23}}{E_{22}} & \frac{1}{E_{33}(1-d_3)} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{G_{12}(1-d_4)} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{G_{23}(1-d_5)} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{G_{13}(1-d_6)} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{13} \end{bmatrix} \quad (32)$$

263 where  $\varepsilon_{ii}$  and  $\gamma_{ij}$  (with  $i = 1, 2, 3$  and  $j = 1, 2, 3$ ) are the components of the strain tensor,  $\sigma_{ij}$  are the  
 264 components of the stress tensor;  $E_{ij}$ ,  $\nu_{ij}$  and  $G_{ij}$  are the Young's moduli, Poisson's ratios and shear moduli,  
 265 respectively, and  $d_i$  are damage parameters associated to different failure mechanisms. The evolution of the  
 266 damage variables is controlled by the fracture toughness along each direction, leading to a linear decay once  
 267 the onset of damage has been reached.

### 268 3.1. Failure modelling

269 Failure mechanisms can be either in-plane and out-of-plane. The theoretical model for thin laminates  
 270 presented in Section 2 accounts for in-plane energy absorption mechanisms such as fibre breakage and  
 271 matrix failure, and neglects the contribution of shear plugging and crushing. The proposed 3D numerical  
 272 model, however, includes all the energy absorption mechanisms involved in ballistics to further validate the  
 273 applicability range of the theoretical model (Alonso et al., 2018a, 2020).

274 In-plane tensile and compression fibre failure are triggered once the following criteria are reached Chang  
 275 and Chang (1987):

$$276 \left( \frac{\sigma_{11}}{X_{11i}} \right)^2 + \left( \frac{\sigma_{12}}{S_{12}} \right)^2 + \left( \frac{\sigma_{13}}{S_{13}} \right)^2 = 1 \quad (33)$$

$$\left( \frac{\sigma_{22}}{X_{22i}} \right)^2 + \left( \frac{\sigma_{12}}{S_{12}} \right)^2 + \left( \frac{\sigma_{13}}{S_{13}} \right)^2 = 1 \quad (34)$$

277 where  $X_{lkr}$  and  $S_{lk}$  are the normal and shear failure stresses respectively associated to  $l = 1, 2, 3$  and  
 278  $k = 1, 2, 3$ .  $r = (t, c)$  accounts for tension and compression.

279 Failure by matrix cracking is assumed to be caused by in-plane shear tension according to Hashin (1980),  
 280 leading to

$$\left( \frac{\sigma_{12}}{S_{12s}} \right)^2 = 1 \quad (35)$$

281 where  $S_{12s}$  is the maximum shear strength (Xiao et al., 2007). The through-thickness matrix and fibre  
 282 failure criterion is associated to shear plugging and the onset of failure is given by

$$\left( \frac{\sigma_{13}}{S_{13s}} \right)^2 + \left( \frac{\sigma_{23}}{S_{23s}} \right)^2 = 1 \quad (36)$$

283 The crush failure criterion is associated to compression along the thickness of the laminate, as described by

$$\left( \frac{\sigma_{33}}{X_{33}} \right)^2 = 1 \quad (37)$$

284 Finally, the interlaminar damage model used in the finite element analyses is based on the classical cohesive  
 285 zone method by means of a traction-separation law (Turon et al., 2007). Damaged is assumed to initiate  
 286 when the following criterion is met,

$$\left( \frac{\langle t_n \rangle}{N} \right)^2 + \left( \frac{t_s}{S} \right)^2 + \left( \frac{t_t}{S} \right)^2 = 1 \quad (38)$$

287 where  $t_n$ ,  $t_s$  and  $t_t$  are the normal and shear stresses, respectively, and  $N$ ,  $S$  are the damage threshold normal  
 288 and shear strengths of the cohesive elements. The Benzeggagh-Kenane (BK) fracture criterion (Kenane and  
 289 Benzeggagh, 1997) governs the evolution of damage after the onset of failure,

$$\Gamma^C = \Gamma_n^C + (\Gamma_s^C - \Gamma_n^C) \left( \frac{\Gamma_s + \Gamma_t}{\Gamma_n + \Gamma_s + \Gamma_t} \right)^\eta \quad (39)$$

290 where  $\Gamma_n^C$  and  $\Gamma_s^C$  are defined as the critical energy release rates for delamination in modes *I* and *II*, which  
 291 correspond to pure tension and shear mode, respectively.  $\Gamma_n$ ,  $\Gamma_s$  and  $\Gamma_t$  account for the work dissipated due  
 292 to the displacements along the normal and shear directions, caused by normal and shear stresses, respectively.  
 293  $\eta$  is a characteristic parameter of the BK law that accounts for the increase in toughness with the mode  
 294 mixity (Abaqus6.14, 2014).

295 Further details about the failure modelling can be found in Alonso et al. (2020).

### 3.2. FE Implementation

The finite element simulations were carried out using Lagrangian 3D elements. The dimensions of the plates were the same than the experimental tests (150x150 [mm<sup>2</sup>]). To simulate the plates, reduced integration solid elements were used (Abaqus C3D8R). Cohesive elements were used to simulate the joint between the plies (Abaqus COH3D8). A convergence study was carried out to ensure the validity of the results. The mesh was divided in two regions, the impact zone and its surroundings, where the density of elements is higher leading to a finer mesh, and the rest, where the density of elements decay gradually as long as the region is further away from the impact zone resulting into a coarser mesh. A number of 10 and 20 solid elements were used along the thickness direction for 3 mm and 6 mm thick specimens, respectively. For the cohesive elements, 1 element (0.001 mm thick) was used along the thickness direction to simulate the joint between two plies. The projectile (density 7800 kg.m<sup>-3</sup> and diameter 7.5 mm) was simulated as a spherical analytical surface and an exponential decay friction model was used to simulate the friction between projectile and target. Regarding the boundary conditions, the projectile was constrained to only move through the thickness direction and the exterior borders of the laminate were clamped. The elements were removed when any of the damage variables reached the value of 1. Further details of the finite element model implementation can be found in Alonso et al. (2020).

## 4. Results and discussion

This section describes the results obtained with the theoretical model and compares these with the finite element analysis. The theoretical model is first validated with residual velocities obtained experimentally. Secondly, both models are used to analyse the influence of geometrical characteristics and material properties on the protective capability of thin woven laminates against ballistic impacts.

### 4.1. Validation of the theoretical model

The theoretical model is validated with experimental results from the ballistic response of E-glass/Polyester woven laminates of 3 and 6 mm thickness (Buitrago-Pérez et al., 2010). The laminates tested were clamped in a steel frame. Then, a 7.5 mm steel projectile was propelled against the laminate. Helium was used in a pressurised chamber to propel the impactor.

A comparison between the predicted theoretical, numerical and experimental residual velocities is shown in Figure 3. The theoretical ballistic limit is the impact velocity at which the residual velocity is different from zero. Note that for the determination of the experimental and numerical ballistic limit, the Lambert-Jonas equation was used to adjust the experimental and numerical data curves (Lambert and Jonas, 1976),

$$v_r = A(v_i^p - v_{bl}^p)^{\frac{1}{p}} \quad (40)$$

where  $A$  and  $p$  are empirical parameters.

Good agreement with the theoretical model is observed for both the ballistic limit and the absorbed energy. The thinner laminate performs better, with the highest error (10%) obtained for the ballistic limit of the 6 mm thickness plate. The inversion of the curves for the thicker laminates (Figure 3) can be explained as follows. This model is based on the assumption that the laminate behaves as a membrane, bending and accelerating, and this assumption is proven to work very well for 3 mm. Nevertheless, 6 mm is very close to the threshold found by (Alonso et al., 2018a) from which a laminate can be considered thick. Consequently, since this model, as Figure 9 shows, gives an essential importance to energy-absorption mechanisms such as elastic deformation of fibres (a part of FF) or acceleration of the laminate ( $A$ ), which are in fact higher in the ballistic limit because the contact time is maximum, it does not reproduce so accurately the behaviour for 6 mm thick panels because the bending and elastic deformation of fibres is more difficult to appreciate when the thickness increases. Therefore, since bending and thus acceleration of the laminate and elastic deformation of fibres absorb less energy, the model overestimates the energy absorbed and so the ballistic limit.

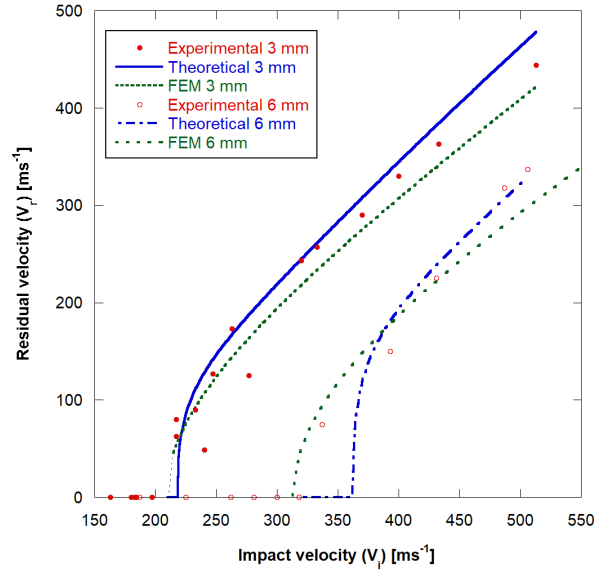


Figure 3: Comparison of experimental Buitrago-Pérez et al. (2010) and numerical results with the proposed theoretical model.

340 To fully validate the theoretical model it is necessary to check if the simplification of the wave theory  
341 on which the model is based is reasonable. We considered that the laminate is homogeneous and thus  
342 we used a longitudinal wave velocity with the homogenized properties of the laminate. To validate this  
343 hypothesis, the propagation of the longitudinal wave was measured in the FE model and compared to the  
344 simplified theoretical prediction. Two cases near the ballistic limit were analysed; 3 mm and 6 mm thickness  
345 plates impacted at velocities of  $240 \text{ m.s}^{-1}$  and  $337 \text{ m.s}^{-1}$  respectively. The theoretical longitudinal radius  
346 prediction was calculated as the longitudinal wave velocity times the time while the FEM prediction was  
347 calculated analysing the integration point of an element at a desired instant. By the time the stress is  
348 different from zero in the integration point, the longitudinal wave was considered to reach the element.  
349 Figure 4 shows the comparison between the theoretical and FE predictions of the longitudinal wave radii for  
350 the two cases analysed. An almost perfect agreement is observed between theoretical and FE predictions for  
351 both cases. It means that the assumption related to the longitudinal wave velocity in the theoretical model  
352 is valid.

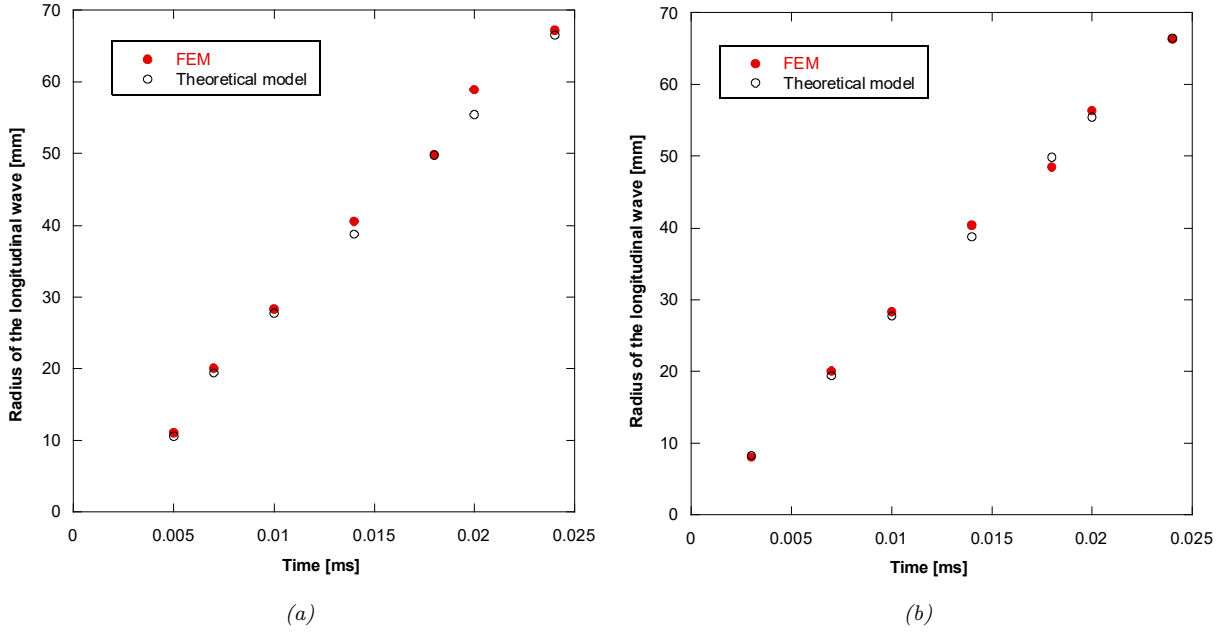


Figure 4: Comparison of the theoretical and numerical predictions of the longitudinal wave radii for (a) a 3 mm thick laminate subjected to an impact velocity of 240m.s<sup>-1</sup> (b) a 6 mm thick laminate subjected to an impact velocity of 337m.s<sup>-1</sup>.

#### 353 4.2. Analysis of the $\Pi$ groups of the problem

354 The performance of the proposed theoretical model is assessed through a number of analysis of the  $\Pi$   
 355 groups of the problem, the influence of the laminate thickness and projectile diameter on the ballistic limit.  
 356 The corresponding results are shown in Figure 5a for thickness-to-diameter ratios ( $\Pi_e$ ) of 0.85, 1 and 1.15. As  
 357 expected, the predicted ballistic limit increases with the ratio  $\Pi_e$ . The three cases analysed present the same  
 358 behaviour. Figure 5a shows that the ballistic limit decreases when increasing the projectile diameter even  
 359 though the thickness increases in the same magnitude since  $\Pi_e$  remains constant. Eventually the ballistic  
 360 limit tends to an asymptote in the range analysed for thin laminates. To explain these results, Figure 5b  
 361 shows the projectile mass grows faster than the laminate mass. As a consequence, the projectile penetrates  
 362 easier into the laminate due to the fast growth of the impactor kinetic energy. The shape of the curves  
 363 showed in Figure 5 is the same suggesting that the ballistic limit is governed by the mass laminate-mass  
 364 projectile ratio.

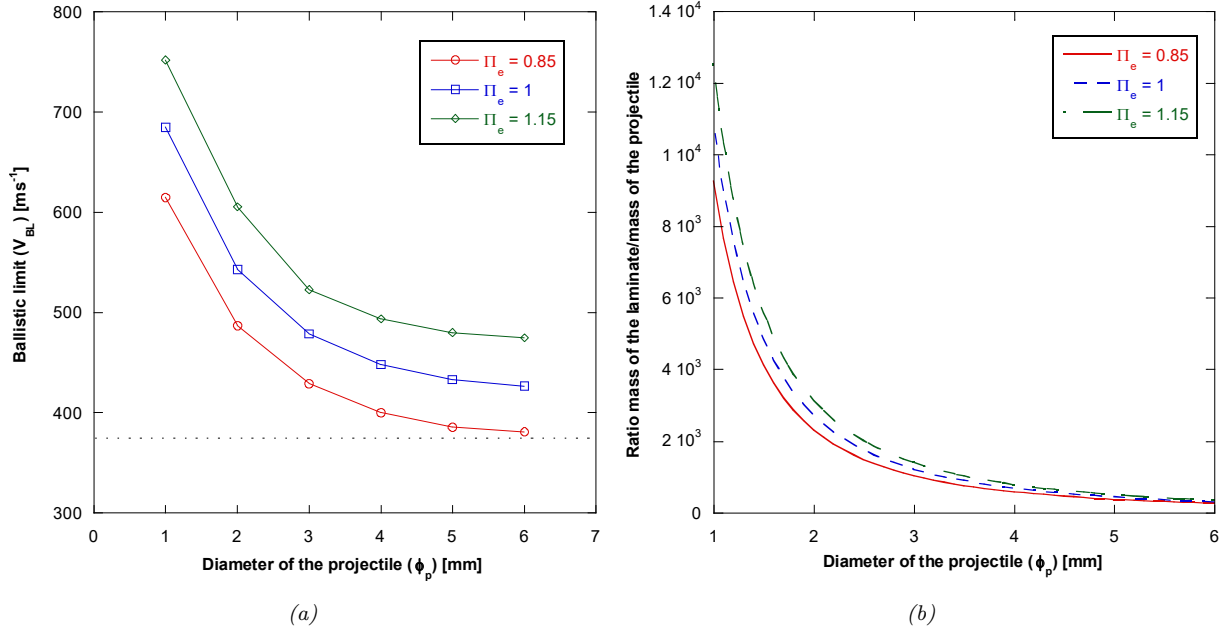


Figure 5: (a) Theoretical model ballistic limit predictions for different thickness-to-diameter ratios ( $\Pi_e = 0.85$ ,  $\Pi_e = 1$  and  $\Pi_e = 1.15$ ) (b) Mass of the laminate / mass of the projectile ratios for different thickness-to-diameter ratios ( $\Pi_e = 0.85$ ,  $\Pi_e = 1$  and  $\Pi_e = 1.15$ ).

365 The results in Figure 6 show the influence of the Young's modulus in direction 11 ( $E_{11}$ ) on the ballistic  
366 limit of the laminate, for thicknesses of 3 and 6 mm. The ballistic limit increases with the  $E_{11}$ , with this  
367 effect becoming more evident for the higher thickness laminate. Thicker laminates absorb more energy and  
368 thus the ballistic curves of Figure 6b are shifted to the the right compared to Figure 6b. The ballistic limit  
369 grows with  $E_{11}$  because the energy absorbed by elastic deformation and tensile failure of fibres, which are  
370 the most important energy-absorption mechanisms near the ballistic limit, increases following Eqs. (22),  
371 (19) show. Therefore, the results in terms of the ballistic limit are physically consistent since the laminate  
372 capability to stop the projectile is expected to be greater if the in-plane stiffness grows. Moreover, all the  
373 curves collapse into one for higher velocities. This result can be explained by the fact that, if the impact  
374 velocity tends to infinite, the absorption capability of the laminate tends to zero regardless of the material  
375 properties.

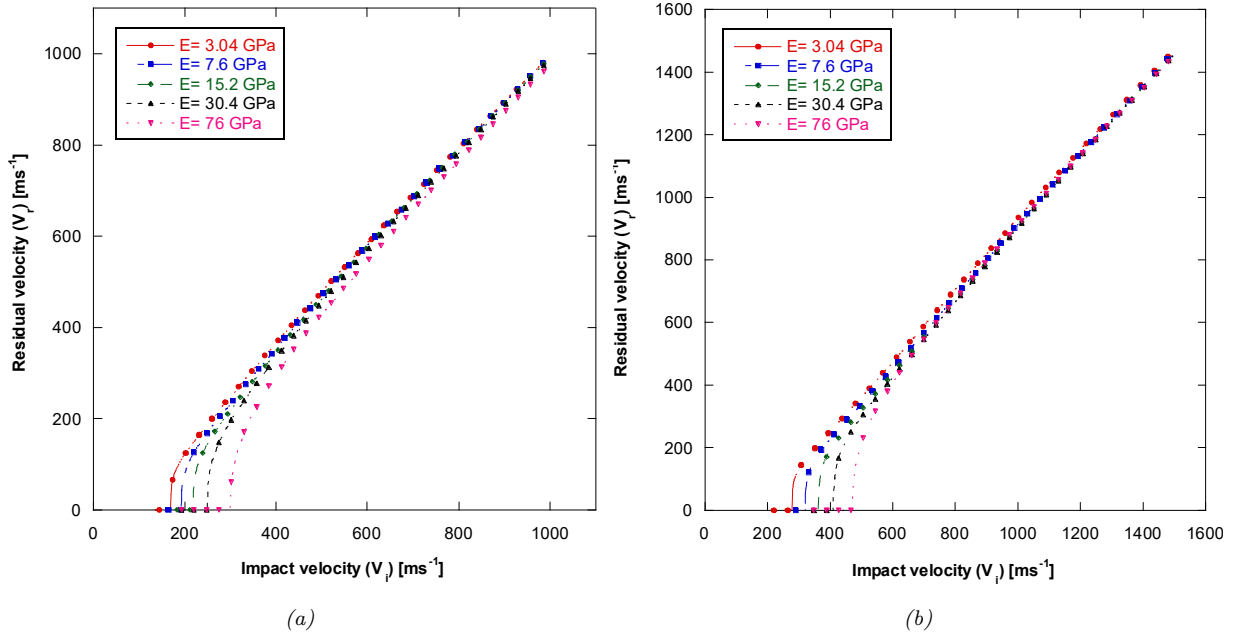


Figure 6: Residual velocity curves for the (a) 3 mm and (b) 6 mm laminates: Influence of the Young's modulus in direction 11 in the energy absorption capacity of the laminate.

#### 376 4.3. Plate penetration

377 Penetration is evaluated for laminates with 3 and 6 mm thickness, and three ballistic regimes, below the  
 378 ballistic limit, where no complete penetration occurs (197 and 262 m.s<sup>-1</sup> respectively), at the ballistic limit  
 379 (240 and 337 m.s<sup>-1</sup> respectively) and above the ballistic limit (320 and 487 m.s<sup>-1</sup> respectively).

380 The theoretical prediction of the penetration rate  $\bar{\delta}$  is validated with numerical simulations. This is a  
 381 way to validate the hypothesis of the relative displacement formulated in the theoretical model by means  
 382 of the phenomenological function  $\bar{k}(\tau, \bar{v}(\tau))$ . In the numerical model, the penetration is calculated as the  
 383 relative displacement between the centre of mass of the projectile,  $x(t)$ , and the rear face of the laminate,  
 384  $x_l(t)$ . Figure 7 illustrates an instant of time from where the projectile does not interact anymore  
 385 with the laminate for a full penetration case in a FE simulation.

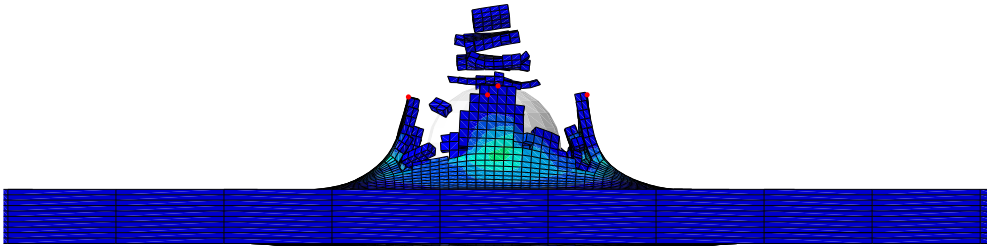


Figure 7: Instant of time in one of the numerical simulations from where the projectile does not interact anymore with the laminate.

386 The observed correlation between numerical and theoretical penetration values is shown in Figure 8 for  
 387 the laminates with 3 and 6 mm thickness, respectively. Simulations have been conducted at velocities below  
 388 (Figures 8a, 8d), near (Figures 8b, 8e) and above (Figures 8c, 8f) the ballistic limit. Good agreement  
 389 between theoretical and numerical results is observed for both panels at velocities below the ballistic limit



390 (Figures 8a, 8d). Final penetration predicted by both models matches as shown in Figures 8a, 8d. A good  
 391 enough agreement is observed for ballistic limit velocities for the laminate with 3 mm thickness (Figure 8b).  
 392 However, for laminates with 6 mm thickness (Figure 8e), the numerical model overestimates the relative  
 393 displacement between both bodies compared to the theoretical model. The differences are caused because  
 394 full penetration occurs in the FE model while the projectile gets stuck in the theoretical model (note that  
 395 the relative displacement does not reach the value of 1 in the theoretical model prediction). Last, Figures 8c,  
 396 8f show a similar prediction of both models for plates subjected to impact velocities above the ballistic limit.  
 397 Nevertheless, the predictions are more similar for the 3 mm thickness case. Overall, the worse agreement  
 398 for thicker laminates can be explained by the study carried out by Alonso et al., 2018. The hypothesis  
 399 of membrane behaviour assumed in this theoretical model works worse with thickness increase, with 6 mm  
 400 being very close to the transition from thin to thick laminate. Therefore, the prediction of parameters related  
 401 to this hypothesis is expected to worsen with thickness increase, as shown in Figure 8. In addition, other  
 402 effects assumed negligible in the theoretical model such as through-thickness failure mechanisms become  
 403 more important and are not properly captured.

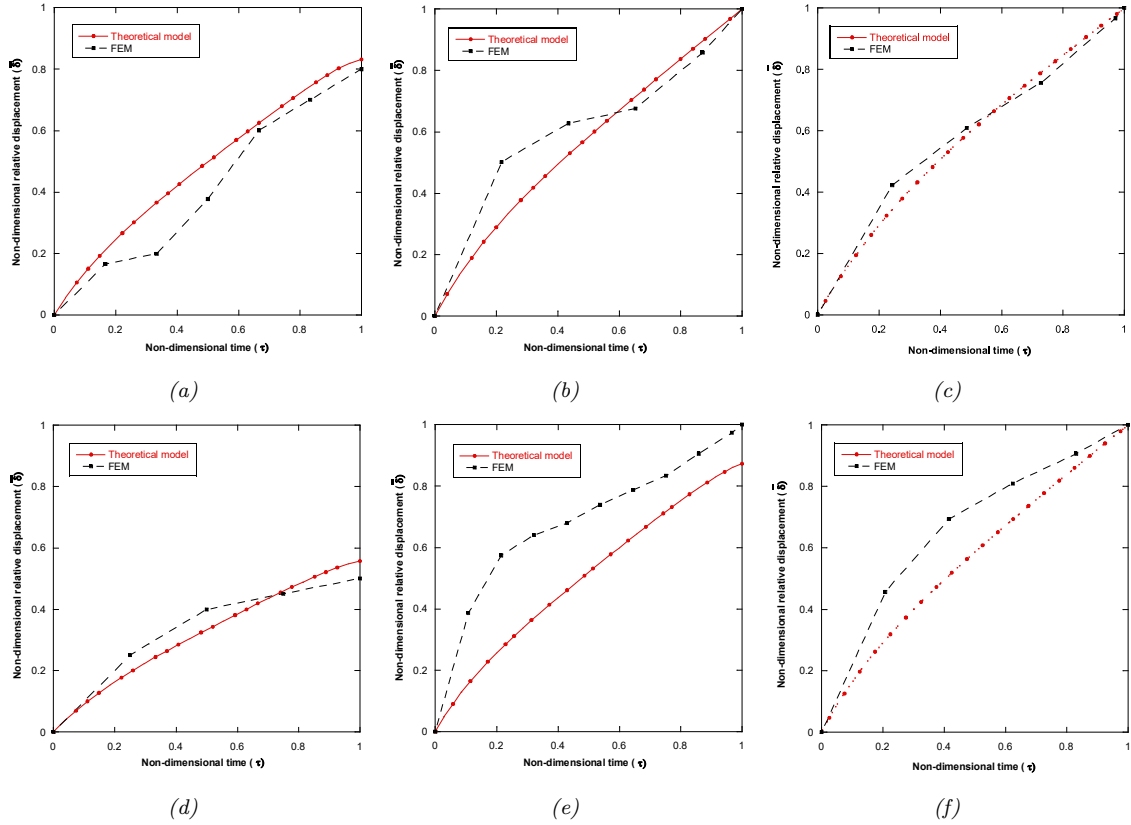


Figure 8: Penetration,  $\bar{\delta}$ , vs non-dimensional time for 3 mm laminates, for impact velocities of (a)  $197\text{m.s}^{-1}$ , (b)  $240\text{m.s}^{-1}$  and (c)  $320\text{m.s}^{-1}$  and for 6 mm laminates, for impact velocities of (d)  $262\text{m.s}^{-1}$ , (e)  $337\text{m.s}^{-1}$  and (f)  $487\text{m.s}^{-1}$ .

#### 4.4. Failure mechanisms and energy absorption

404 A comparison between the numerical and theoretical model predictions of the energy absorbed by each  
 405 individual mechanism is presented in the following paragraphs. The following mechanisms are accounted for  
 406 in the theoretical model: (i) fibre failure (see equations 33 and 34) is associated to elastic deformation and  
 407 tensile failure of fibres; (ii) matrix failure (see equation 37) is associated with matrix cracking; (iii) kinetic  
 408 energy of the elements is associated to kinetic energy of the laminate. Crush failure and shear driven failure  
 409

410 (see equations 35 and 36) are not contemplated in the theoretical model. We assume this hypothesis since it  
411 has been demonstrated that these mechanisms do not play a major role in the penetration process for thin  
412 laminates (Alonso et al., 2020). Note that friction is another energy dissipation process present during the  
413 whole impact process but it is considered negligible under the thin laminates hypotheses and thus it is not  
414 shown in Figure 9 in the FEM part. The numerical energy results are determined using the methodology  
415 proposed by (Alonso et al., 2020).

416 The comparison of the absorbed energy fractions of each mechanism is shown in Figure 9 for the 3 and  
417 6 mm laminates, respectively. This study has been carried out at the same velocities as the penetration  
418 analysis. The main energy-absorption mechanisms observed in this analysis is fibre breakage (corresponding  
419 to elastic deformation and tensile failure of fibres in the theoretical model), being even more important below  
420 the ballistic limit. Matrix cracking and delamination are proved to have a minor role in all the cases analysed  
421 despite the ballistic regime. Nevertheless, the remaining energy is dissipated mostly by through-thickness  
422 failure mechanisms, such as crushing and shear plugging, with higher influence of crushing for velocities  
423 above the ballistic limit. The influence of this mechanism is higher for 6 mm laminates since the laminate is  
424 closed to be considered as thick Alonso et al. (2018a). At higher velocities, the fibre breakage occurs fast and  
425 thus the energy absorbed by this mechanism is less important. Therefore, the shear and through-thickness  
426 resistances gain importance as shown in Figures 9c, 9f. Consequently, fibre failure and compression are  
427 the most important energy-absorption mechanisms for 6 mm laminates. The kinetic energy absorbed by  
428 the laminate acceleration increases its importance when full penetration occurs since the laminate keeps  
429 moving at the moment of complete penetration as Figures 9c, 9f show. Note that the energy absorbed by  
430 the laminate acceleration is recoverable.

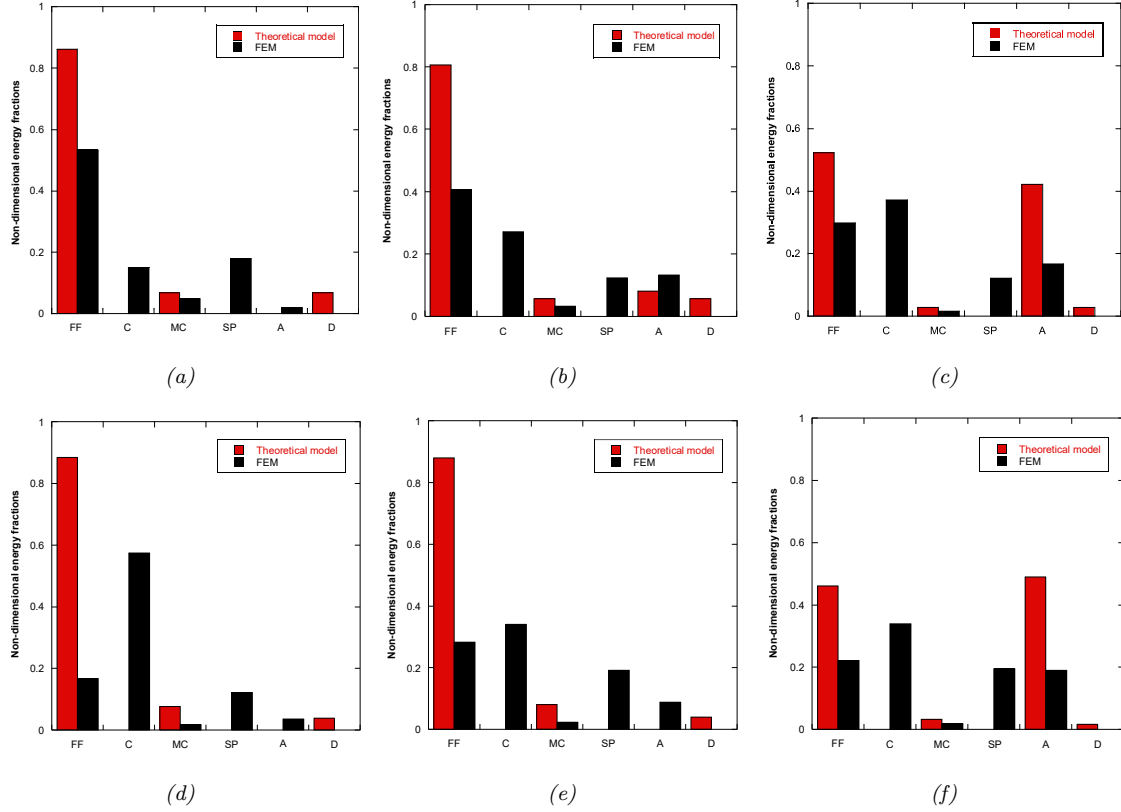


Figure 9: Theoretical and numerical non-dimensional energy fractions from left to right: Fibre failure (FF), compression (C), matrix cracking (MC), shear plugging (SP), acceleration of the laminate (A) and delamination (D) for a 3 mm thick specimen subjected to velocities of (a) 197m.s<sup>-1</sup>, (b) 240m.s<sup>-1</sup> and (c) 320m.s<sup>-1</sup> and for a 6 mm thick specimen subjected to velocities of (d) 262m.s<sup>-1</sup>, (e) 337m.s<sup>-1</sup> and (f) 487m.s<sup>-1</sup>.

431 Overall, both theoretical and FE predictions are consistent in terms of the relative roles of energy-  
 432 absorption mechanisms. Note that the FE model accounts for a wide variety of energy absorption mecha-  
 433 nisms. Despite this fact, the main trends observed in both theoretical and FE approaches are consistent.  
 434

#### 435 4.5. Comparison with flat-ended projectile

436 The versatility of the theoretical model allows for a change in the impactor shape. The impact behaviour  
 437 of 3 mm and 6 mm thick specimens subjected to flat-ended projectile impacts with the same diameter and  
 438 mass (7.5 mm and 1.725 g) is studied. Since there are not experimental data available, the main objective  
 439 of this section is to compare the influence of the projectile shape in the theoretical model predictions. To  
 440 carry out this study, the projected area of the projectile has to be changed, therefore equation 12 becomes  
 441 in equation 41 and all the equations in which  $\bar{\phi}(\bar{\delta})$  is involved change accordingly,  
 442

$$\bar{\phi}(\bar{\delta}) = \frac{\phi_p}{e} \quad (41)$$

443  
 444 Figure 10 shows a comparison between the predicted theoretical residual velocities for the two thicknesses  
 445 and projectile shapes analysed.

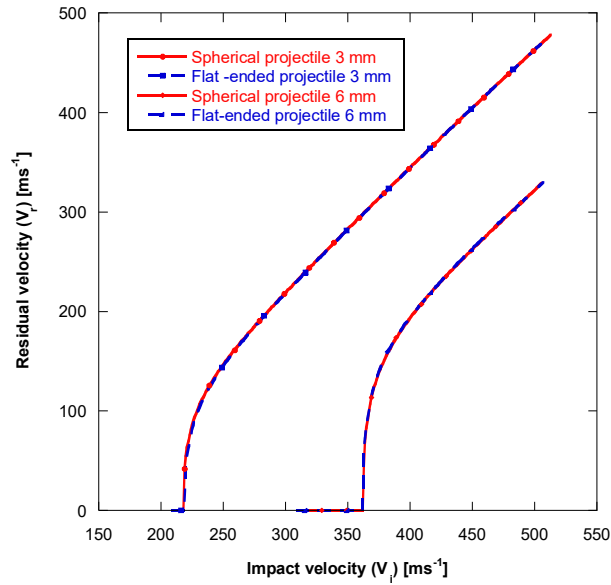


Figure 10: Comparison between the predicted theoretical ballistic response for 3 mm and 6 mm laminates impacted by spherical and flat-ended projectiles.

446 Good agreement between the two shapes is observed. Actually, the curves almost overlap. From this, it  
 447 can be inferred that the projectile shape has not an important influence on the ballistic response as long as  
 448 the diameter and mass remain constant, which is in agreement with previous works (Ulven et al., 2003).

449 Another important point to check is if the hypotheses for thin laminates are met when changing the  
 450 projectile shape. To accomplish that, the predicted theoretical and numerical relative displacement versus  
 451 time for velocities below the ballistic limit are shown in Figure 11 for the two thicknesses.

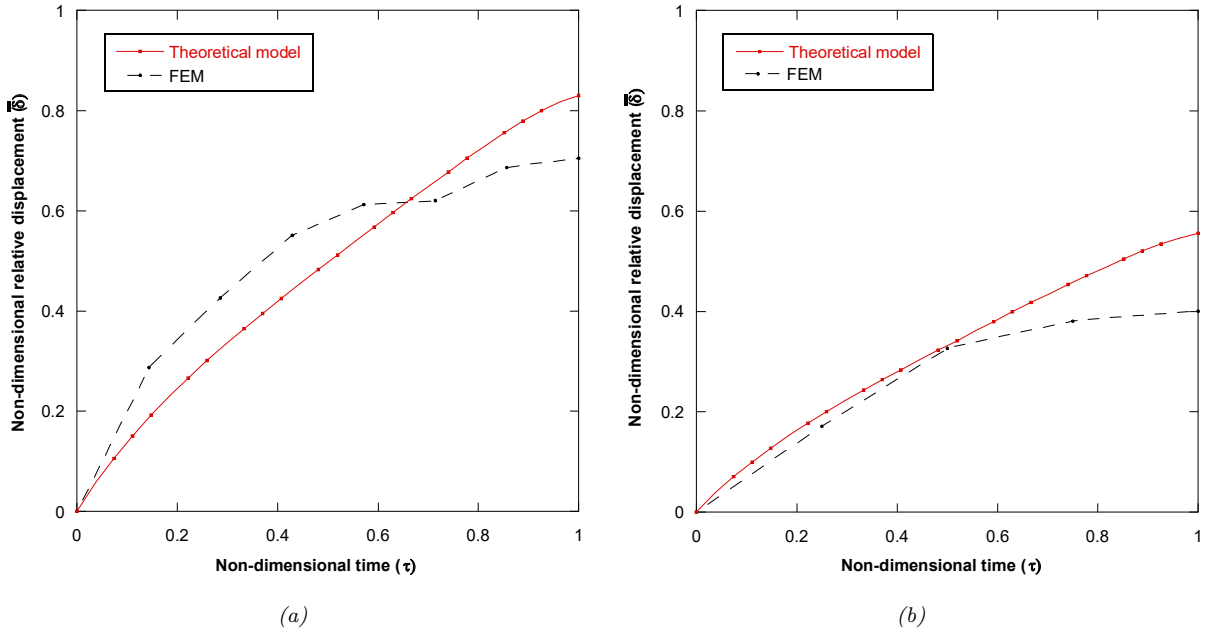


Figure 11: Flat-ended projectile penetration,  $\bar{\delta}$ , vs non-dimensional time for 3 mm laminates, for an impact velocity of (a) 197m.s<sup>-1</sup>, and for 6 mm laminates, for an impact velocity of (b) 262m.s<sup>-1</sup>.

452 Figure 11 shows a good agreement between theoretical and numerical results for both thicknesses. Never-  
 453 theless, the difference in the predicted final penetration observed in Figure 11b is higher than in Figure 11a.  
 454 Again, this is a consequence of the laminate thickness since the higher the thickness, the worse the suitability  
 455 of the thin laminate hypotheses considered.

456 Figure 12 shows a comparison between the two models' prediction by means of the relative importance  
 457 of the energy-absorption mechanisms for the same two velocities.

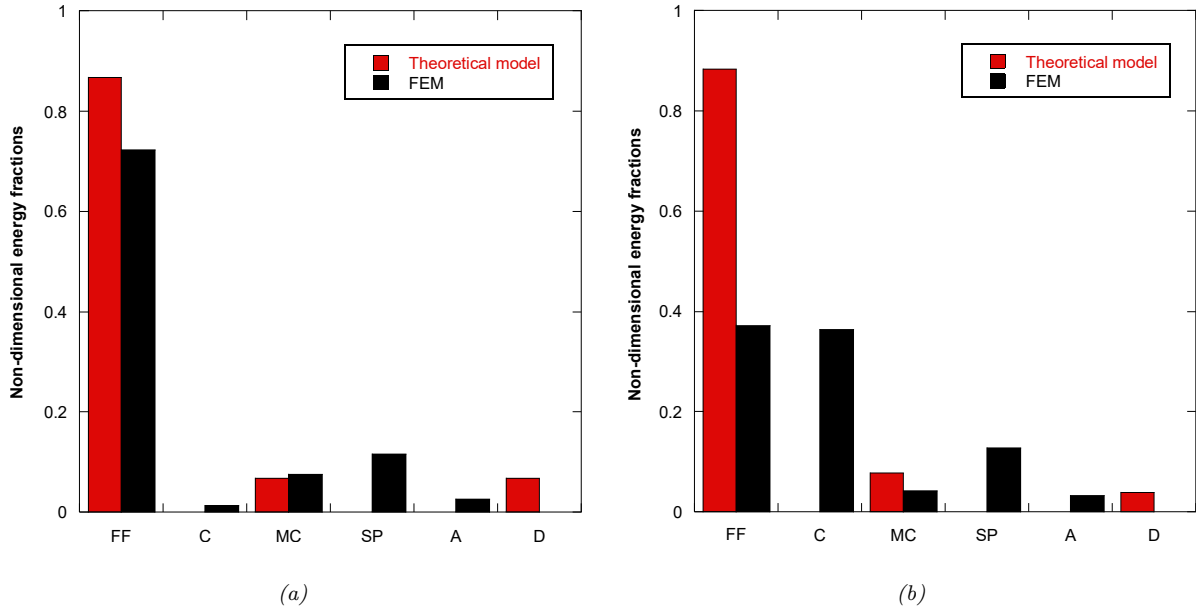


Figure 12: Theoretical and numerical non-dimensional energy fractions from left to right: Fibre failure (FF), compression (C), matrix cracking (MC), shear plugging (SP), acceleration of the laminate (A) and delamination (D) for a specimen impacted by a flat-ended projectile with an impact velocity of (a)  $197\text{m.s}^{-1}$  for 3 mm thickness and (b)  $262\text{m.s}^{-1}$  for 6 mm thickness.

Overall, the relative importance of the terms is the same than predicted in Figures 9a, 9d. Fibre failure is the most important energy-absorption mechanism and out-of-plane energy-absorption mechanisms increase with thickness. Nevertheless, in Figure 12b fibre failure and compression share the dominant role while in Figure 9d compression is significantly more important. It makes sense that fibre failure is more important for flat-ended projectiles since the whole diameter is in contact from the beginning contributing to fibre breakage. Therefore, it can be inferred that, although the ballistic response is almost the same, the relative contribution of the energy-absorption mechanisms/failure modes can change with the projectile shape.

## 5. Conclusions

The ballistic impact of thin woven E-glass fibre/polyester composites was studied in this paper, and a new theoretical model was proposed to describe the mechanical response and penetration of the laminates. The non-dimensional energy-based theoretical model considers traditional energy absorption mechanisms from previous models, and incorporates new hypotheses. The main governing equation of the problem is a non-linear second-order differential equation on the position of the projectile with respect to the laminate front face in the initial configuration, which can be solved by numerical integration. In order to validate the theoretical model, the results of the ballistic limits as well as the residual velocities were compared to the experimental and the numerical model results. Agreement between theoretical model and experimental and FE results was found, with a maximum difference lower than 10%.

Two representative II groups are studied to assess the physical consistency of the model. These results suggest that the ballistic response is governed by the laminate/projectile and the laminate Young's modulus  $E_{11}$ .

The FE model was used to validate some of the hypotheses of the theoretical model. Overall, the predictions of relative displacement or penetration,  $\bar{\delta}$ , by both models are in good agreement. The most critical cases, when full penetration does not occur, present small differences in the predictions for both thicknesses.

481 This agreement becomes worse when increasing the laminate thicknesses, as its structural response experi-  
482 ences a transition from thin to thick laminate. For thin laminates, changing the projectile shape leads to  
483 the same ballistic results if diameter and mass are maintained. However, the relative contribution of the  
484 energy absorption mechanisms/failure modes may change with projectile geometry.

485 The failure mechanisms in the numerical model are associated to the different energy-absorption mech-  
486 anisms considered in the theoretical model. Energy-absorption mechanisms are compared between the the-  
487 oretical and the FE models for three ballistic regimes: below, above and at the ballistic limit for for 3 mm  
488 and 6 mm laminates. Fibre failure is identified as the most important energy-absorption mechanism while  
489 matrix cracking and delamination are proved to play a minor role. Out-of-plane failure mechanisms such  
490 as compression and shear plugging are more important at high-impact velocities and for 6 mm laminates,  
491 which are close to the thick behaviour.

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