



30th International Conference on Flexible Automation and Intelligent Manufacturing (FAIM2021)
15-18 June 2021, Athens, Greece.

Evaluation of T-joints in aluminium structures under different geometries

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Abstract

The adhesive bonding technique is nowadays very popular in industrial applications, and is gradually replacing other more traditional bonding methods (fastened, welded and riveted joints) due to some advantages. However, its application supposes accurate methods for strength prediction. As a result, the techniques to predict the strength of adhesive joints has highly evolved. The eXtended Finite Element Method (XFEM) is a recent innovation implemented over the (Finite Element Method) FEM that enables crack growth to be modelled. However, its application to adhesive joints is still scarce. This work consists of an experimental and XFEM analysis of aluminium alloy T-joints, adhesively-bonded with three adhesive types. A parametric study is undertaken regarding the curved adherends' thickness (t_p), with values between 1 and 4 mm. The adhesives Araldite[®] AV138 (strong but brittle), Araldite[®] 2015 (less strong but moderately ductile) and the Sikaforce[®] 7752 (with the smallest strength but highly ductile) were tested. A comparative analysis between the different joints conditions was undertaken by plotting peel (σ_y) and shear (τ_{xy}) stresses. The XFEM predictive capabilities were tested with different damage initiation criteria. It was found that, provided that the modelling conditions are properly set, accurate numerical results can be found.

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Keywords: Fracture; Finite element analysis; eXtended Finite Element Method; Bonded joint.

1. Introduction

The use of adhesive bonds greatly increased in industrial applications, as they have multiple advantages compared to other more traditional bonding methods (fastened, welded and riveted joints). The aeronautical, naval, automotive and aerospace industries are good examples where adhesive joints are widely applied. More uniform distribution of stresses, ease of manufacture, possibility of joining different materials and low cost are the main advantages of adhesive bonds. The main disadvantages are related to the requirement of surface preparation, low peel strength and difficulties in quality control and safety. The most common adhesive joint configurations are single-lap joints (SLJ), double-lap joints (DLJ) and scarf joints [1]. SLJ are the most common. However, they develop major σ_y peak stresses. DLJ are more difficult to manufacture but, on the other hand, σ_y stresses greatly diminish. Scarf joints are

highly efficient when compared to SLJ because of the reduction of stress concentrations [1]. Although these types of joints are the most used in the industry, other types of joints have specific applications. Stepped-lap configurations can be used in composite joining due to the easiness to make the step design during the materials' fabrication process [2]. T-joints find application in the naval and aeronautical industries. In the naval industry, they allow joining panels with the hull [3] and the fiberglass hull with anti-flood panels [4]. In the aeronautical industry, they are used to join wing panels and fuselage sections [5]. Several works were carried out to evaluate T-joints, using either analytical or numerical techniques [6, 7].

The number of approaches to predict the strength of adhesive joints has increased over the years. Actually, analytical and numerical techniques have become more refined and with higher accuracy. The FEM allows modelling complex geometries with precision, due to computational advancements

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10.1016/j.promfg.2020.10.119

and Computer Aided Engineering (CAE) tools. The use of Continuum Mechanics supposes using the obtained stresses or strains, whose maximum values are used in appropriate failure criteria to assess failure. However, this technique has limited applicability because of stress singularities (which make the predictions dependent on the applied mesh) and neglecting of fracture mechanics concepts [8]. Actually, in a bonded joint FEM analysis, stresses near the singular regions increase with the mesh refinement, making convergence impossible [9]. Traditional Fracture Mechanics-based techniques can be applied to the study of the behaviour of structures that contain defects, such as cracks. These cracks can result from stress concentrations, usually located in holes, notches or interfaces between different materials. However, it is not mandatory that the structures to be analysed already have cracks, which is a limitation of this method [10]. Cohesive Zone Models (CZM) were developed to describe damage under static loads in the cohesive process zone around the crack tip. They are based on cohesive elements, which allow connecting solid elements of two-dimensional (2D) and three-dimensional (3D) structures, using pre-established traction-separation laws [11]. CZM were tested and optimized to promote structural damage initiation and crack propagation simulations on cohesive and interfacial fracture problems, and delamination in composites. The use of CZM to model structures enables to create one or more regions or interfaces in which damage nucleation and growth is made possible by the softening and release of homologous nodes of the cohesive elements [12]. FEM simulations based on continuum mechanics wrongly consider that the solid elements undergo plasticization without taking damage. Damage mechanics simulations work by inducing damage to the elements through the reduction of transmitted loads between solid elements. Thus, it is possible to perform the simulation of crack growth, in which the cracks can assume a pre-defined trajectory or an arbitrary trajectory within a finite region [8]. In Damage Mechanics, a damage parameter is established to cause a change in the response of the constituent materials through the depreciation of the strength or stiffness, as occurs in adhesive layers, or in composite delaminations, to model damage during loading [13, 14]. The insertion of a damage variable in the constitutive law of the material enables simulating damage before and after crack nucleation. Two types of damage variables can be introduced in the models: variables that empirically depreciate the properties of the materials, without any relation to the damage mechanism, and variables that have a physical significance, by directly relating to the observed type of damage (for example the size of porosities or micro-cavities) [15]. The growth of damage is usually ruled by the load function for static simulations [16] and as a function of the number of cycles for fatigue modelling [17, 18]. The XFEM is a recent variant of the FEM to model damage growth in structures, although it is yet seldom studied within the context of bonded joints. This method uses damage laws to predict fracture, based on strength concepts to infer damage initiation of damage and deformations for failure. Comparing the XFEM with CZM, the XFEM has the clear advantage of not requiring the crack to follow a predefined path by the user. This is because crack propagation occurs freely inside the material, without the geometry of the discontinuities

being coincident with the mesh or the necessity to correct the mesh in the crack vicinity [19]. The XFEM is based on the concept of partition of unity, and its implementation in the FEM can be accomplished by introducing local enrichment functions for the displacements near the crack tip, allowing damage to grow and respective separation between the cracked faces [20]. Mubashar et al. [21] carried out a study on the damage and failure modelling of adhesively-bonded SLJ with spew fillets at the overlap ends, combining two methods: XFEM (to perform the modelling of the crack in the fillet region where the crack path is unknown) and CZM (applied to model crack progression and damage along the adhesive bond interface). The numerical analysis was performed in Abaqus®. Aluminium alloy 2024 T3 adherends were bonded with the epoxy adhesive FM73-M, and the adhesive was modelled with elastoplastic properties, obtained in tensile tests. This work allowed to conclude that the XFEM is capable of predicting, with a high degree of precision, the crack onset location and path within the spew fillet. Moreover, it is possible to combine the XFEM with CZM to more accurately predict crack initiation and growth in bonded joints, including at the interface. However, this technique is limited by the existence of a potential discontinuity in the crack at the XFEM-CZM transition. Stuparu et al. [22] conducted a study on the combined use of CZM and XFEM for the strength prediction of bonded joints. The SLJ configuration was tested, with aluminium adherends and the adhesive Araldite® AV138. The following parameters were used: adherends' thickness (t_p) of 5 mm, adhesive thickness (t_A) of 1, 3 and 5 mm, overlap length (L_0) of 20 mm and sample width (B) of 25 mm. The numerical analysis was done in Abaqus®. The XFEM was used to simulate failure within the adhesive, considering a strain criterion (less mesh dependent than stress criteria) for crack onset prediction. Thus, crack initiation/propagation will always take place orthogonally to the maximum principal strains. On the other hand, CZM was equated to simulate an interfacial failure between the adhesive and adherends. Different t_A and the positions of initial bonding flaws were tested, resulting in modifications of the XFEM crack trajectories, eventually attaining the interface. It was shown that the use of XFEM is well complemented by CZM to promote crack growth after the XFEM crack attained the interface.

This work consists of an experimental and XFEM analysis of aluminium alloy T-joints, adhesively-bonded with three adhesive types. A parametric study is undertaken regarding t_{p2} , with values between 1 and 4 mm. The adhesives Araldite® AV138 (strong but brittle), Araldite® 2015 (less strong but moderately ductile) and the Sikaforce® 7752 (with the smallest strength but highly ductile) were tested. A comparative analysis between the different joints conditions was undertaken by plotting σ_y and τ_{xy} stresses. The XFEM predictive capabilities were tested with different damage initiation criteria.

2. Experimental work

2.1. Adherends and adhesives

The T-joints are made of three AW6082 T651 aluminium alloy aluminium adherends bond together. This is a high-

strength alloy, characterized in a previous work [23]. Fig. 1 shows typical stress-strain (σ - ϵ) curves of this aluminium alloy, whose relevant properties in bulk tensile testing are: Young's modulus (E) of 70.1 ± 0.8 GPa, tensile yield stress (σ_y) of 261.7 ± 7.7 MPa, tensile strength (σ_f) of 324.0 ± 0.2 MPa and tensile failure strain (ϵ_f) of $21.7 \pm 4.2\%$.

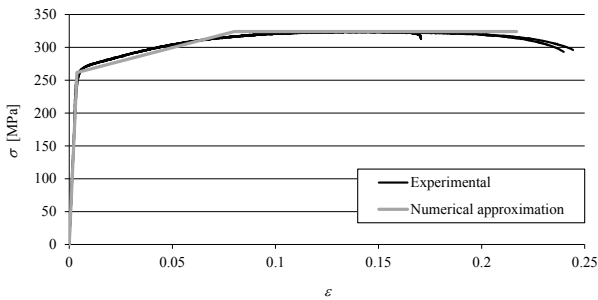


Fig. 1. Experimental and numerical σ - ϵ curves of the aluminium.

The following adhesives were tested in the T-joint configuration: Araldite® AV138 (brittle epoxy), Araldite® 2015 (ductile epoxy) and the Sikaforce® 7752 (high-elongation polyurethane). All adhesives were formerly tested and the respective properties detailed in references [23, 24]. The tensile mechanical properties (E , σ_y , σ_f and ϵ_f) resulted from bulk tests to dogbone specimens. The fabrication process for these specimens followed the indications stipulated in the NF T 76-142 French standard. The shear mechanical properties (Shear modulus – G , shear yield stress – τ_y , shear strength – τ_f and shear failure strain – γ_f) were estimated by Thick Adhering Shear Tests (TAST). In this case, the 11003-2:1999 ISO standard was considered for fabrication and testing protocols. The TAST specimens were made with DIN C45E steel adherends, and curing was undertaken in a rigid mould to guarantee that the cured specimens are aligned [23].

Table 1. Properties of the adhesives Araldite® AV138, Araldite® 2015 and Sikaforce® 7752 [23, 24].

Property	AV138	2015	7752
Young's modulus, E [GPa]	4.89±0.81	1.85±0.21	0.49±0.09
Poisson's ratio, ν	0.35 ^a	0.33 ^a	0.30 ^a
Tensile yield stress, σ_y [MPa]	36.49±2.47	12.63±0.61	3.24±0.48
Tensile failure strength, σ_f [MPa]	39.45±3.18	21.63±1.61	11.48±0.25
Tensile failure strain, ϵ_f [%]	1.21±0.10	4.77±0.15	19.18±1.40
Shear modulus, G [GPa]	1.81 ^b	0.70 ^b	0.19 ^b
Shear yield stress, τ_y [MPa]	25.1±0.33	14.6±1.3	5.16±1.14
Shear failure strength, τ_f [MPa]	30.2±0.40	17.9±1.8	10.17±0.64
Shear failure strain, γ_f [%]	7.8±0.7	43.9±3.4	54.82±6.38
Toughness in tension, G_{IC} [N/mm]	0.20 ^c	0.43±0.02	2.36±0.17
Toughness in shear, G_{IIC} [N/mm]	0.38 ^c	4.70±0.34	5.41±0.47

^a manufacturer's data

^b Estimated from the Hooke's law using E and ν

^c estimated in Campilho et al.[23]

The toughness properties of the adhesives were estimated with the Double-Cantilever Beam (DCB) test (tensile fracture

toughness or G_{IC}) and the End-Notched Flexure (ENF) test (shear fracture toughness or G_{IIC}). Table 1 gives an overview of the obtained data, which will be used in this work for input in the numerical simulations. To be noted that the values of yield stress were defined considering a plastic strain of 0.2%.

2.2. Experimental details

Fig. 2 represents the geometry and dimensions of the T-joints. The relevant dimensions are the following: $L_0=25$ mm, $B=25$ mm, base length $L_T=80$ mm, base thickness $t_{P1}=3$ mm, $t_{P2}=1, 2, 3$ and 4 mm, L-part length $L_A=60$ mm, L-part radius $R=5$ mm and $t_A=0.2$ mm.

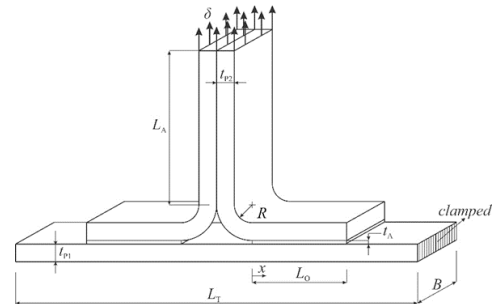


Fig. 2. Representative geometry and dimensions of the T-joints.

Specimen fabrication was initiated by cutting/bending the adherends to the respective shapes. The straight adherend, used as the specimen base, is obtained by cutting, in an automated cutter, a bar to the final dimensions. The L-parts were subjected to an identical procedure, but then they were manually bent in a press, such that the end surfaces were at an angle of 90° , and applying a tool with the chosen R , to produce the geometry depicted in Fig. 2. The surface preparation before bonding consisted of manually increasing the roughness by sanding using emery paper with coarse grain (60 grit), followed by wiping with cleaning agent (acetone) to eliminate dust and oxides, thus achieving a strong bond. Joining of the different parts was accomplished in a jig that positioned the three adherends in the layout of Fig. 2 and enabled keeping this position throughout the entire adhesive hardening process. To assure the specified t_A , steel spacers with identical thickness to t_A were inserted at the edges of the bonded parts are removed after adhesive curing. Before placing the spacers, they were initially coated with Loctite® Frekote 770NC demoulding agent, to guarantee easy removal after the adhesive has cured, which is essential to prevent damage to the cured adhesive layers. With the specimens in position after depositing the adhesive, and with the spacers providing the correct offset between adherends, grips were used to apply pressure to the set and enable curing to take place. This process was accomplished during a one-week period. The final step consisted of trimming the excess cured adhesive by milling. As a result of this set of operations, it was possible to obtain a good representation of the theoretical geometry of Fig. 2, with emphasis to the bondline end geometry positioned at $x/L_0=0$ (x is the horizontal coordinate initiating at the bondline end). The specimens were tested as depicted in Fig. 2, i.e. by clamping the edges of the

straight adherend and pulling in peel while transversely restraining the upper joint edge. This was done in a Shimadzu AG-X 100 testing machine, equipped with a 100 kN load cell, at an approximate temperature of 20°C and testing speed of 1 mm/min. Five specimens were fabricated and tested for each joint type. A minimum of 4 valid tests was always assured for each joint type.

3. Numerical work

3.1. Numerical simulation details

The analysis performed in Abaqus® was 2D, considering a geometrically non-linear FEM formulation. The aluminium adherends (both base and L-parts) were modelled as solids with the plastic behaviour defined in a previous work for the same material [25]. The adhesive layer was also modelled with solid elements, but with enriched XFEM formulation. The XFEM model is presented in the next Section. In all cases, 4-node solid elements with plane-strain conditions were used (with Abaqus® reference CPE4). A perfect adhesion was considered in the models between the adherends and adhesive, since the models were built as a single part with different partitions and materials. Two types of meshes were applied: a more refined mesh to perform a stress analysis in the elastic domain, such that the stress plots are accurate, and a less refined mesh to promote the XFEM failure analysis. Fig. 3 shows an example of mesh refinement for a T-joint with $t_{p2}=1$ mm, with details at the loaded overlap edge for the stress and failure analyses.

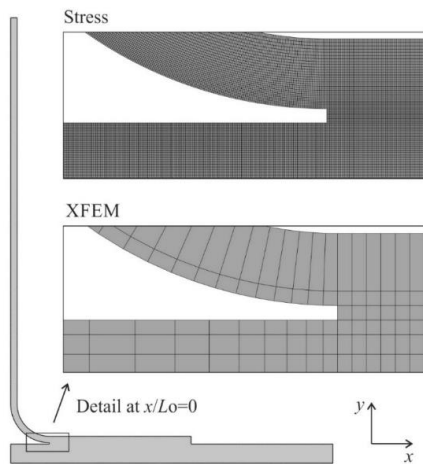


Fig. 3. Mesh details at $x/L_0=0$ for the joint with $t_{p2}=1$ mm: stress analysis and XFEM strength prediction analyses.

Mesh bias effects were employed in the models to grade the elements' size, enabling to reduce the total number of elements but without compromising the accuracy. This was done by considering a more refined mesh near the adhesive layer and towards its edges, and a coarser mesh in the zones with less stress variations [26]. The elements' side dimensions in the adhesive layer for the XFEM simulations were 0.2×0.2 mm along the bondline, i.e., only one element through-thickness was considered. On the other hand, the models for the stress

analysis were comprised of 0.02×0.02 mm elements at the overlap edges. The boundary conditions consisted of fixing the base edges to simulate gripping in the testing machine, applying symmetry at the middle of the specimen and pulling the curved adherends' edges in peel.

3.2. XFEM formulation

As an extension to the conventional FEM, the XFEM is based on the integration of enrichment functions in the FEM formulation [27]. These functions allow modelling the displacement jump between crack faces that occur during the propagation of a crack. The Abaqus® XFEM formulation enables the user to create a pre-crack, or it can initiate cracks in un-cracked regions by using initiation criteria. In this last scenario, considered in this work, damage initiates and subsequently propagates during the simulation at regions experiencing stresses and/or strains higher than the corresponding limiting values. Six crack initiation criteria are available in Abaqus®. The MAXPS (maximum principal stress) and MAXPE (maximum principal strain) criteria are based on the introduction of the following functions (by the respective order)

$$f = \left\{ \frac{\langle \sigma_{\max} \rangle}{\sigma_{\max}^0} \right\} \quad \text{or} \quad f = \left\{ \frac{\langle \varepsilon_{\max} \rangle}{\varepsilon_{\max}^0} \right\}. \quad (1)$$

σ_{\max} and σ_{\max}^0 represent the current and allowable maximum principal stress. The Macaulay brackets indicate that a purely compressive stress state does not induce damage. ε_{\max} and ε_{\max}^0 represent the current and allowable maximum principal strain. Crack growth for the MAXPS and MAXPE criteria is software defined as orthogonal to the maximum principal stress/strain direction. As a result of this, and due to the inherent mixed-mode loading of these joints, the crack grows fast towards the adherends. For these two criteria, the maximum load (P_m) estimation was thus considered to take place at the time of first cracking in the adhesive layer. The MAXS (maximum nominal stress) and MAXE (maximum nominal strain) criteria are represented by the following functions, respectively.

$$f = \max \left\{ \frac{\langle t_n \rangle}{t_n^0}, \frac{t_s}{t_s^0} \right\} \quad \text{or} \quad f = \max \left\{ \frac{\langle \varepsilon_n \rangle}{\varepsilon_n^0}, \frac{\varepsilon_s}{\varepsilon_s^0} \right\}. \quad (2)$$

t_n and t_s are the current normal and shear traction components to the cracked surface. t_n^0 and t_s^0 represent the respective limiting values. The strain parameters have identical significance. The quadratic nominal stress (QUADS) and quadratic nominal strain (QUADE) criteria are based on the introduction of the following functions, respectively

$$f = \left\{ \frac{\langle t_n \rangle}{t_n^0} \right\}^2 + \left\{ \frac{t_s}{t_s^0} \right\}^2 \quad \text{or} \quad f = \left\{ \frac{\langle \varepsilon_n \rangle}{\varepsilon_n^0} \right\}^2 + \left\{ \frac{\varepsilon_s}{\varepsilon_s^0} \right\}^2. \quad (3)$$

For the MAXS, MAXE, QUADS and QUADE criteria the user can select between horizontal or vertical crack growth (in

this work horizontal growth, i.e., along the adhesive layers' length, was selected). All the six aforementioned criteria are fulfilled, and damage initiates, when f reaches unity. For damage growth, the fundamental expression of the displacement vector \mathbf{u} , including the displacements enrichment, is written as [28].

$$\mathbf{u} = \sum_{i=1}^N N_i(x) [\mathbf{u}_i + H(x)\mathbf{a}_i] \quad (4)$$

$N_i(x)$ and \mathbf{u}_i relate to the conventional FEM formulation, corresponding to the nodal shape functions and nodal displacement vector linked to the continuous part of the formulation, respectively. The second term between brackets, $H(x)\mathbf{a}_i$, is only active in the nodes for which any relating shape function is cut by the crack and can be expressed by the product of the nodal enriched degree of freedom vector including the mentioned nodes, \mathbf{a}_i , with the associated discontinuous shape function, $H(x)$, across the crack surfaces. The method is based on the establishment of phantom nodes that subdivide elements cut by a crack and simulate separation between the newly created sub-elements. Propagation of a crack along an arbitrary path is made possible by the use of these phantom nodes that initially have exactly the same coordinates than the real nodes and that are completely constrained to the real nodes up to damage initiation. After being crossed by a crack, the element is partitioned in two sub-domains. The discontinuity in the displacements is made possible by adding phantom nodes superimposed to the original nodes. When an element cracks, each one of the two sub-elements will be formed by real nodes (the ones corresponding to the cracked part) and phantom nodes (the ones that no longer belong to the respective part of the original element). These two elements that have fully independent displacement fields replace the original one. Thus, the crack size increment for a given crack orientation is equal to the distance between the cracked element's edges. From this point, each pair of real/phantom node of the cracked element is allowed to separate according to a suitable damage law up to failure. At this stage, the real and phantom nodes are free to move unconstrained, simulating crack growth. The parameters introduced in Abaqus[®] were taken from Table 1. A linear softening XFEM law was considered with an energetic failure power law criterion of the type

$$\left(\frac{G_I}{G_{IC}}\right)^\alpha + \left(\frac{G_{II}}{G_{IIC}}\right)^\alpha = 1, \quad (5)$$

in which α is the damage law exponent ($\alpha=1$ for linear softening).

4. Results

4.1. Experimental failure modes

All failures took place beginning with crack propagation at $x/L_0=0$ and growing towards the other edge. After failure, the fracture surfaces were inspected and cohesive failures were found for all adhesives and t_{p2} . However, in some cases,

especially for the joints bonded with the Araldite[®] AV138, failure sometimes took place near to one of the adherend/adhesive interfaces Fig. 4 shows an example for the joints with $t_{p2}=2$ mm), such that visually it resembled an adhesive failure. However, careful surface inspection including optical microscope observations revealed that the adherends that at first hand suffered from an adhesive failure actually were covered by a thin layer of adhesive. These findings are consistent with previous observations on this particular adhesive [25]. The fracture surfaces for the joints bonded with the Araldite[®] 2015 and Sikaforce[®] 7752 were smoother, indicative of ductile fractures, with a clearer evidence of cohesive failures. L-part adherend plasticization was detected in all joints bonded with the Araldite[®] 2015 and $t_{p2}=1$ mm, and also with the Sikaforce[®] 7752 and $t_{p2}=1$ and 2 mm, although for this last case it was under 0.1%.

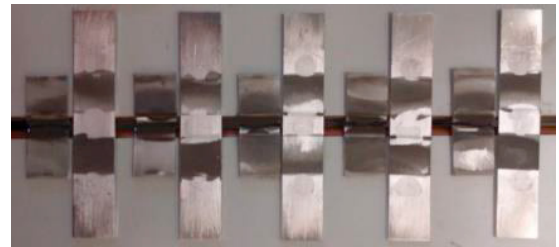


Fig. 4. Fracture surfaces for the five sets of T-joints bonded with the Araldite[®] AV138 and with $t_{p2}=2$ mm (for each set the L-parts are at the left and the base adherend to the right).

4.2. Stresses in the elastic domain

This Section briefly evaluates σ_y and τ_{xy} stresses along the adhesive during the initial stages of the elastic loading, such that the differences between adhesives and t_{p2} are properly accounted for. The graphics are represented as follows:

- The x -axis represents the normalized distance along the adhesive length (x/L_0), in which x is the horizontal coordinate beginning at the leftmost edge of the adhesive layer, i.e. the edge closest to the central portion of the joint;
- The y -axis represents the normalized σ_y and τ_{xy} stresses (σ_y/τ_{avg} and τ_{xy}/τ_{avg} , respectively). τ_{avg} is the average τ_{xy} stress along the adhesive mid-thickness for the respective t_{p2} .

Between the three adhesives, stresses in the elastic loading stage are identical, with the sole difference residing in the normalized peak values, since these increase in proportion with the adhesive stiffness (E values compared in Table 1) [29]. Thus, graphically, only the results for one adhesive are presented, in this case for the Araldite[®] 2015 (Fig. 5), which has the middle stiffness. It can be first observed that σ_y stresses are prevalent over τ_{xy} stresses, which was expected due to the expected peel loading, although τ_{xy} of non-negligible magnitude also develop due to the sliding of the L-part over the base adherend. Independently of t_{p2} , peak stresses or at least stress perturbations exist at the vicinity of $x/L_0=0$ and 1 mm, which consist of the stress singularity regions.

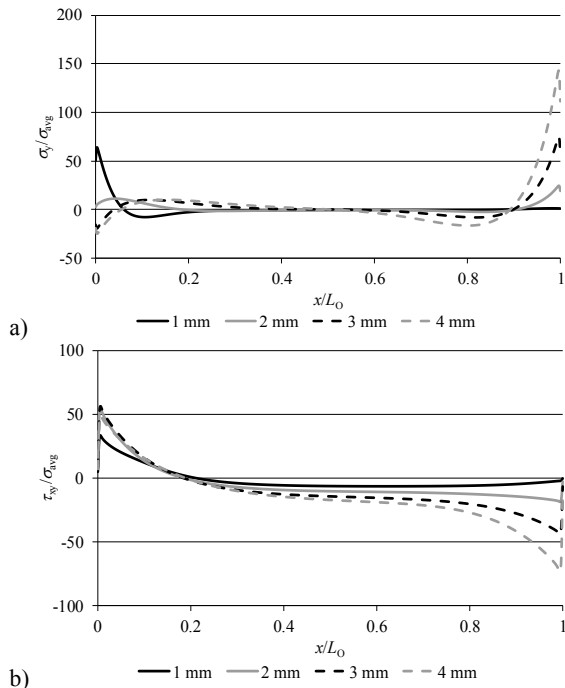


Fig. 5. Normalized σ_y (a) and τ_{xy} (b) stresses along the adhesive mid-thickness for the T-joints bonded with the Araldite® 2015.

σ_y stresses were identical between the three adhesives, although with the aforementioned differences in magnitude of peak stresses. The curves depicted in Fig. 5 (a), relating to the Araldite® 2015, showed intermediate peak values between the other two adhesives. For the joints with $t_{p2}=1$ mm, the critical region was undoubtedly $x/L_0=0$, due to the small stiffness of the L-part, arising from its low thickness. For the other t_{p2} , σ_y stresses are either close to zero or compressive at this location. The increase of t_{p2} gradually reduces the harmful effect of σ_y peak stresses at $x/L_0=0$, but gradually loads the other edge ($x/L_0=1$), which is deemed to occur because of the growing thickness and stiffness of the L-part. Actually, this change in deformation pattern of the L-part, namely the significant reduction of its bending, works against the natural curvature developing in the straight adherend due to the tensile pulling. This gives rise to tensile σ_y peak stresses near $x/L_0=1$, while the other edge tends to become lightly loaded. This effect can even be responsible by an alteration of the failure path if t_{p2} reaches a given limit. On the other hand, σ_y stresses tend to span for a bigger region of the bonding length with the increase of t_{p2} , which can be responsible for a P_m improvement.

Identically to that found for σ_y stresses, the behaviour of τ_{xy} stresses has large similarities between adhesives, although with peak magnitude variation between adhesives due to their stiffness differences. The T-joints bonded with the Araldite® 2015, whose curves are shown in Fig. 5 (b), presented a middle behaviour between the three tested adhesives. The smallest τ_{xy} peak stresses are those of the joints with $t_{p2}=1$ mm and, for this joint, τ_{xy} stresses peak close to $x/L_0=0$, while being negligible for the remainder of the bonding portion. On the other hand, the increase of t_{p2} lightly increases τ_{xy} peak stresses near $x/L_0=0$, concurrently with a progressive increase of τ_{xy} peak

stresses near $x/L_0=1$. This modification in the τ_{xy} stress plots is related to the aforementioned behaviour of σ_y stresses, i.e., due to the stiffening effect of the L-part, which is not accompanied by the transverse deformation of the base adherend, which subsequently leads to large adhesive strains near $x/L_0=1$. Moreover, the variation of τ_{xy} stress plots can also influence the failure onset and growth, identically for that discussed for σ_y stresses.

4.3. Experimental strength

The experimentally obtained P_m for the T-joints bonded with the three adhesives are presented in this Section. Fig. 6 reports the average P_m vs. t_{p2} curves, including the standard deviation of the experiments.

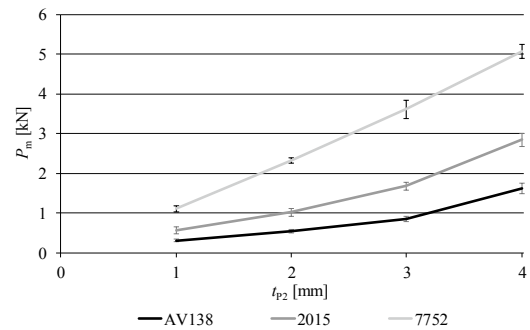


Fig. 6. Experimental P_m vs. t_{p2} curves for the three adhesives.

It can be found that P_m always increases, and by a large amount, with t_{p2} , irrespectively of the adhesive. The two Araldite® adhesives show an increasing growth with t_{p2} , while the Sikaforce® 7752 has a marked linear evolution of the P_m vs. t_{p2} curve. The percentile P_m increase for the T-joints with $t_{p2}=2, 3$ and 4 mm, over the T-joint with $t_{p2}=1$ mm, was respectively of 75.0%, 173.8% and 419.3% (Araldite® AV138), 81.2%, 197.8% and 403.7% (Araldite® 2015) and 110.9%, 227.1% and 358.6% (Sikaforce® 7752). This marked P_m improvement with t_{p2} is mainly due to the σ_y stress levelling effect that is visible in Fig. 5 (a) near $x/L_0=0$, which corresponds to the stress initiation site. Between adhesives, the Sikaforce® 7752 clearly outperforms the other two adhesives, despite being the less strong amongst the three adhesives (Table 1). The P_m improvement of the T-joints bonded with this adhesive, over those bonded with the Araldite® AV138, ranges between 213.8% ($t_{p2}=4$ mm) and 328.0% ($t_{p2}=2$ mm). On the other hand, compared against the joints bonded with the Araldite® 2015, the P_m improvement varies between 78.5% ($t_{p2}=4$ mm) and 128.1% ($t_{p2}=2$ mm). It is also visible in Fig. 6 that P_m for the joints bonded with the Araldite® 2015 are higher than those with the Araldite® AV138. In fact, depending on t_{p2} , P_m may almost double that obtained with the Araldite® AV138 adhesive, although this adhesive has higher strengths. The improvement is minimum for $t_{p2}=4$ mm (78.4%) and maximum for $t_{p2}=3$ mm (97.2%). These results show that, in a test geometry that is mainly loaded in peel, and which promotes stresses to be concentrated in small areas, flexible and ductile adhesives behave best. In one hand, the flexibility tends to

increase the area along which stresses are being transferred (Fig. 5). On the other hand, the ductility permits plasticization of the adhesive at the stress concentration sites while the regions in the vicinity becomes loaded, resulting in an overall improved behaviour.

4.4. Numerical evaluation of the XFEM initiation criterion

The XFEM initiation criteria described in Section “XFEM formulation” are evaluated against the experimental data, by directly comparing P_m with the experiments. At this stage, the linear energetic criterion, considering $\alpha=1$ in equation **Erro! A origem da referência não foi encontrada.**, is considered in all simulations. As it was previously discussed, the use of the MAXS, MAXE, QUADS and QUADE criteria results in crack onset and growth parallel to the adhesive layer, while MAXPS and MAXPE criteria leads to cracking perpendicular to the maximum principal stresses or strains, which subsequently makes the crack grow in the direction of the adherends. Fig. 7 shows the P_m comparison for all t_{p2} between the different XFEM initiation criteria and the experiments for the adhesives Araldite® AV138 (a), Araldite® 2015 (b) and Sikaforce® 7752 (c).

For the Araldite® AV138, the QUADS and MAXS criteria are closest to the experimental points, and the respective curves are practically overlapped in the figure. Averaged over the experiments, the maximum relative deviations were +8.8% ($t_{p2}=2$ mm) and +9.5% ($t_{p2}=2$ mm), respectively, for these two criteria. The MAXPS criterion revealed to be unsuited, in the manner that it was used, since P_m highly underestimates the tests (up to -82.5% for $t_{p2}=1$ mm). Oppositely to this behaviour, the strain-based criteria (QUADE, MAXE and MAXPE) overshoot the experimental data, with emphasis to the QUADE and MAXE. The highest offsets for these criteria were +113.3% (QUADE), +117.3% (MAXE) and +31.3% (MAXPE), in all cases for $t_{p2}=2$ mm.

Qualitatively, the P_m predictions for the Araldite® 2015 by the six criteria agree with those of the Araldite® AV138. Thus, the QUADS and MAXS criteria are quite close to the experimental values, with a negligible difference between them. The maximum P_m deviations were, in both cases, obtained for $t_{p2}=4$ mm, attaining -8.9% (QUADS) and -8.6% (MAXS). The MAXPS criterion was offset up to -81.3% ($t_{p2}=1$ mm), whilst the maximum deviations for the strain-based criteria attained maximums of +110.7% for both QUADE and MAXE ($t_{p2}=1$ mm), and +55.2% the MAXPE criterion ($t_{p2}=2$ mm).

Finally, the results for the Sikaforce® 7752 were much alike those of the former adhesives although, in this case, even the QUADS and MAXS criteria showed bigger variations to the experimental P_m (up to -13.3% for the QUADS, considering $t_{p2}=4$ mm, and +13.3% for the MAXS, considering $t_{p2}=1$ mm). Identically, the curves for these two criteria overlap. Due to the aforementioned approximations, the MAXPS criterion showed P_m values much below the expected, with a maximum deviation of -85.8% ($t_{p2}=3$ mm). The strain-based criteria significantly over predicted P_m , in line with the previous adhesives. The maximum offsets, all by excess, were +197.8%

for the QUADE ($t_{p2}=2$ mm), +214.7% for the MAXE ($t_{p2}=4$ mm) and +160.5% for the MAXPE ($t_{p2}=2$ mm).

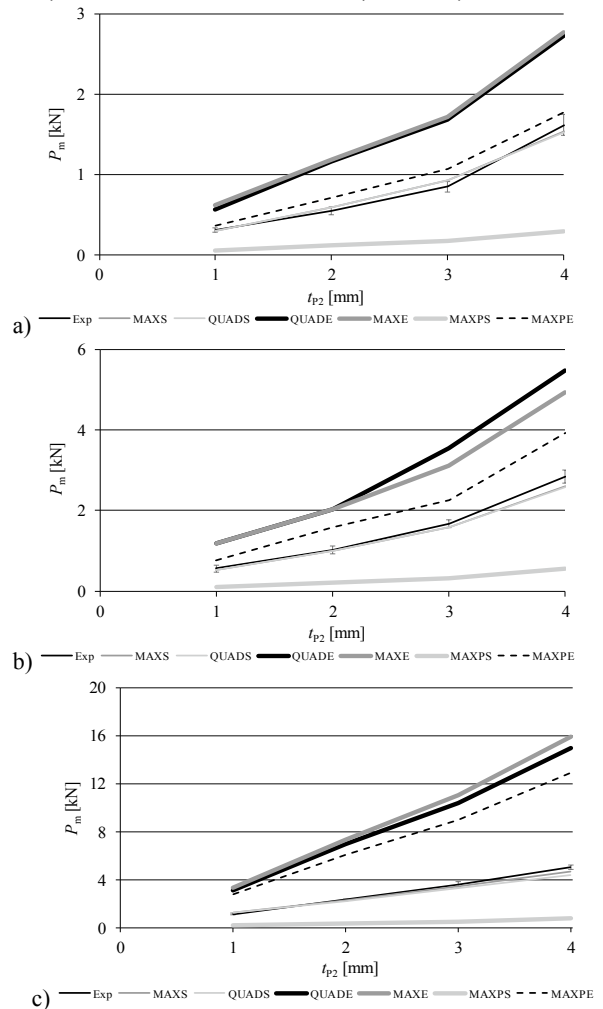


Fig. 7. Experimental and numerical P_m comparison, considering different XFEM initiation criteria, for the T-joints bonded with the Araldite® AV138 (a), Araldite® 2015 (b) and Sikaforce® 7752 (c).

It was shown in a previous work [23] that damage initiation is ruled by the adhesive layer’s stresses rather than the strains (which also vary by a large amount between adhesives). On the other hand, using strain-based criteria can result in major deviations to the real joint behaviour, with an over prediction tendency. This is why the QUADS and MAXS criteria generally work very well. The QUADE and MAXE criteria, being based on strains, naturally present wrong P_m results and should not be considered in the design process of bonded joints. The MAXPS and MAXPE criteria, due to its intrinsic formulation, are unable to promote a realistic damage growth path, since the crack growth direction is ruled by the maximum principal stresses or strains, in a sense that crack initiates and grows perpendicularly to the principal directions. As a result, adherend detaching by the adhesive through all the adhesive layer is rendered unfeasible because the mixed-mode loading induced in the adhesive results in short crack growth in the adhesive before the crack hits the adherend interface. Since P_m

was assessed by the damage initiation load, i.e., at the time the first crack appears in the model, the results do not match the real joint behaviour. Moreover, since stresses and strains in FEM modelling are traditionally mesh dependent [11], the direct use of initiation criteria to assess failure, i.e., without failure criteria, should promote mesh-dependent P_m .

5. Conclusions

This work presented an experimental and numerical assessment of the behaviour of adhesively-bonded T-joints between aluminium adherends, considering different geometric conditions (t_{p2}) and adhesives with different characteristics with respect to the strength and ductility. The experimental analysis showed that, for the particular joints conditions tested, i.e., a predominantly peel loading with major peak stresses, the most ductile although less strong Sikaforce® 7752 is the one that presents better results for all t_{p2} . Increasing t_{p2} highly increased P_m for all adhesives. The difference between adhesives was clarified by a σ_y and τ_{xy} stress analysis being performed to the adhesive layer, which showed that the σ_y peak stresses ruling the failure process are inversely proportional to the adhesives' stiffness. Thus, the corresponding stress plots are more uniform and enable spreading the loads more evenly, which highly benefits P_m . Adding to this, the Sikaforce® 7752 is highly ductile, thus permitting the adhesive to undergo plasticization at the highest stresses zones, whilst lightly loaded regions increase load transfer, before failure. The XFEM analysis applied to the initiation criterion enabled to conclude that, for all the adhesives, the QUADS and MAXS criteria were the most adequate. The MAXPS criterion was inadequate, in view of the simplification taken to estimate P_m . All strain-based criteria (QUADE, MAXE and MAXPE) overshoot P_m by a large amount for the three adhesives, and should not be considered as well.

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