

Prediction and numerical simulation of residual stress in multi-pass pipe welds

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ABSTRACT

A numerical simulation procedure is presented to predict residual stress states in multi-pass welds in oil transportation pipes. In this paper, a two-dimensional thermo-mechanical finite element model is used to calculate the temperature distribution, hardness, and the distribution of residual stresses during multi-pass welding of pipes of dissimilar metals and varying thicknesses. In this model, the temperature dependence of the thermal and mechanical properties of the material was considered. The present model was validated using the hardness measurement. Good agreement was found between the measurement and the numerical simulation results. The simulated result shows that the two-dimensional model can be effectively used to simulate the hardness test and predict the residual stress in the pipe weld. The simulation results and measurements suggest that the model with moving heat source can obtain a good prediction of residual welding stress. Both the two-dimensional and the three-dimensional modeling can be used to estimate the residual stresses in different weld regions and help saving time.

KEYWORDS

numerical simulation, welding pipe, residual stresses, hardness test

1. INTRODUCTION

Welding is a reliable and efficient metal joining process used in almost all industries. Welded joints are extensively used in oil and natural gas pipeline, power plant and pressure vessels. Dissimilar pipe welds are commonly found in various industrial applications due to both technical and economic reasons [[1\]](#page-4-0). One of the significant problems in pipe welded structures is residual stresses caused by shrinkage of the material after the welding process. This is due to the large temperature gradients generated in the weld Fusion Zone (FZ), Heat-Affected Zone (HAZ), and Base Metal (BM). The regions near the weld zone undergo transient heating and cooling cycles during the welding, thus generating plastic deformation and residual stresses in the weld zone. Furthermore, the resulting residual stresses may be responsible for increasing cracking and fatigue in the weld. The measurement of residual stresses developed during welding of the pipe helps in lowering the risk to failure by predicting the effect of residual stresses. The most way to control and analyze residual stresses is to measure them. There are many methods to measure residual stresses are known as nondestructive, semi-destructive and destructive or diffraction as X-ray diffraction, hole drilling and block removal layering [\[2](#page-4-1)]. To reduce the residual stresses effect of welding process quality, research on residual stresses has been a continuous interest among researchers [3[–](#page-4-2)[10](#page-4-2)]. The experimental measurement techniques of residual stresses that involve the complex weld shape or structure, which is very time-consuming and costly. Numerical simulations help to understand and predict residual stresses behaviors of complex welding phenomena in extreme conditions. The prediction of residual stress is essential to optimize the welding process for reducing the stresses effects. Recently, numerical simulation can be used to predict residual analysis stresses and reduce the time and economic costs associated

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with experiments. Finite Element (FE) analysis (numerical simulation) is critical way to help quantify welding mechanism. Welding residual stresses in simulation research, which is based on the finite element analysis has achieved rapid development, and used in weld simulation has been widely used [\[10](#page-5-0)–[18](#page-5-0)]. In this study, the prediction of residual stresses in a dissimilar pipe weld joint made of E355K2 and P460NH_1 is studied by using 2-D finite element method. The simulation hardness of the weld joint is used to validate the model and compare the simulation result with experimental hardness results.

2. MATERIALS AND METHOD

Dissimilar material pipe welds were used in this work because they are commonly found in various industrial applications for both technical and economic reasons. The materials and chemical composition of the pipe used in this study are given below in [Table 1,](#page-1-0) and the welding parameters for weld joints and welding pool shape or Goldak model are given in [Table 2](#page-1-1) [\[19\]](#page-5-1). The filler wire was 3.2 mm in diameter. Multi-pass welding with an inter-pass with maximum temperature value 200 $^{\circ}$ C is used. [Figure 1](#page-1-2) shows the pipe after welding and the 2-D model with fillers was built with (birth and death) element using 8,466 elements. By using appropriate mesh optimization technique, a relatively fine mesh is generated in and around the weld lines and a comparatively coarse mesh is used for areas away from weld line. In numerical simulation analysis, the accuracy of the results and required computing time are determined by the finite element size (mesh density). According to numerical analysis theory, the models with a fine mesh highly accurate results but required longer computing time. On the other hand, the model with coarse mesh (large element size) may lead to less accurate results but not much computing time.

Table 2. Welding parameters

Pass	Current	Voltage	Speed		Welding pool parameters (mm)			
No.	(A)	(V)	mm/s	Efficiency	a	b	c_f	c_r
1	80	23.2	2	0.8		3	5	-8
$\overline{2}$	90	23.6	2	0.8	4	3	5	8
3	100	24	\mathfrak{D}	0.8		3	5	8

3. VALIDATION OF THE 2-D MODEL

In order to set up the accuracy of the numerical model, validation of model is necessary to predict the residual stress. The cross-section welding part was polished after cutting, and the surface of the specimen was etched with HNO3 (nitric acid) 2% solution (nitrate etching agent) to show the passes and FZ and HAZ.

The hardness measurement was carried out from top to bottom with 12 lines, as it shown in [Fig. 2](#page-2-0). The validation of the welding simulation procedure is carried out on a multipass butt weld with eight tracks. The left pipe is P460NH_1

Fig. 1. Pipe after welding and 2-D finite element model with dimension in (mm)

Fig. 2. Pipe after cutting and specimen after hardness measurement

Fig. 3. Temperature profile in dissimilar welds joints

steel (BM1), the right pipe material is E355K2 (BM2) steel and filler metal Böhler is validated. The validation of welding simulation procedure is carried out on a multi-pass butt weld. The results are compared with experimental hardness test. Figure 2 shows measurements with a microscopemounted camera showing an average of the measurement distance from the surface at a given point. The distances are in millimeter, and the hardness values are in HV10. The hardness distribution from the simulation at the weld of the investigated dissimilar material with welding was calculated by using the rule of mixtures. Maynier et al. [\[20\]](#page-5-2) have developed a useful method to predict hardness. The total hardness of steel is calculated dependent on the volume fractions of the constituents of the microstructure:

$$
HV = \begin{pmatrix} FP\% \cdot HV_{F-P} + B\% \cdot HV_B + \\ +M\% \cdot HV_M \end{pmatrix} / 100. \tag{1}
$$

The hardness of the microstructures produced is given by:

$$
HV_M = 127 + 949C\% + 27Si\% + 11Mn\% ++ 16Cr\% Ni\% + 21\log v_R,
$$
\n
$$
HV_B = -323 + 185C\% + 330Si\% + 153Mn\% +
$$
\n(2)

$$
HV_B = -323 + 185C\% + 330Si\% + 153Mn\% ++ 144Cr\% + 191Mo\% + 65Ni\% ++ (logv_R) \left(\begin{array}{c} 89 + 53C\% - 55Si\% - 2Mn\% --20Cr\% - 33Mo\% - 10Ni\% \end{array} \right),
$$

$$
HV_{F-P} = 42 + 223C\% + 53S\%i + 30Mn\% +
$$

$$
HV_{F-P} = 42 + 223C\% + 535\%i + 30Mn\% ++7Cr\% + 9Mo\% + 12.6N\%i++ (log\nu_R) \begin{pmatrix} 10 - 195i\% + 8Cr\% ++4Ni\% + 130V\% \end{pmatrix},
$$
(4)

where: v_R is the cooling rate in K/h ; HV is the hardness (Vickers); X_M , X_B , X_F and X_P are the volume fractions of martensite, bainite, ferrite and pearlite, respectively; HV_M , HV_B , and HV_{F+P} are the hardness of martensite, bainite and the mixture of ferrite and pearlite, respectively. For the calculating of HV_M , Hv_B , and HV_{F+P} the formulas developed by Maynier et al. [[17](#page-5-3)] was used.

[Figure 3](#page-2-1) showed the temperature profile in dissimilar welds joints was validated. The gray region of modeled cross-section denotes fusion zone where the temperature exceeds the melting point $(1,700 \degree C)$, and the color lines indicate the fusion line, HAZ1 and HAZ2 of the weld.

[Figure 4](#page-2-2) gives information about the comparison between the predicted hardness simulation and the actual hardness measurement across weld joints at different regions from the weld centerline near the outer surface. The hardness test in base metal 1 (BM1) showed a steady but significant rise, which leveled out in HAZ1 and remained steady until the border of HAZ2, where a sharp peak occurs. It then falls rapidly to a level that is higher than that in HAZ2. During the distance of welding 30×8 points, as it can be seen, there is good consistency

Fig. 4. Comparison of the hardness measurement with simulation

and acceptable agreement between the simulation and the experimental results.

4. RESULT AND DISCUSSION

4.1. Residual stress results from the 2-D model

4.1.1. Axial residual stress distribution on the outer and outer surface. After ensuring the validity and reliability of the numerical model, [Fig. 5a and b](#page-3-0) show the axial residual stress distribution of the dissimilar weld joint. It can be seen that the residual stress distribution in the weld joint is different due to the presence of two different materials. The maximum axial residual stress under tension is located at the interface of the two metals (weld metal and base metal E355K2) and has a magnitude of -789 MPa, whereas on the weld metal and base metal P460NH_1 side the maximum is around -653 MPa.

4.1.2. Hoop residual stress distribution on the outer and inner surface. [Figure 6a and b](#page-3-1) shows the residual stress distributions along the outer and inner surface. It can be seen that tensile residual stresses are generated on the weld in HAZ1 on the P460NH_1 side in the region between the metal and weld. The tensile stress for E355K2 increases sharply and reaches maximum stress 557 MPa, which is seen along the HAZ and FZ.

Fig. 5. a) Axial residual stress distribution in the weld zone; b) axial stress on the outer and the inner surface

Fig. 6. a) Hoop residual stress distribution on the outer and inner surface from weld centerline; b) Hoop residual stress distribution on the outer and inner surface

4.1.3. Axial and hoop stress distribution through the thick-ness of welding. [Figure 7](#page-3-2) shows the distribution of axial and hoop residual stresses as a function of the thickness of the weld joint. Axial stress remains consistent through the P460NH reducing through the thickness of the weld joint. There is an observed increase in the same through the region of E355K2 metal. However, the increase is not sharp but gradual and is around 50 MPa \pm 50 MPa. The residual hoop stress shows a dramatic increase in the magnitude from the P40NH to E355K2 metal going through the weld thickness.

Fig. 7. Axial and hoop residual stress distribution in the middle

Fig. 8. a) Definition of the middle section with different angles; b) 3D FEM model; c) Axial residual stress in different angle; d) Hoop residual stress in different angle

4.2. Residual stress results from the 3-D model

[Figure 8a](#page-4-3) presented the cross section of 3-D model to investigate the residual stress distribution in different locations. To clarify the feature of residual stress distribution, the middle section of pipe weld shown in Fig. 8b. The final hoop and axial stress distributions of the middle section are shown in Fig. 8c and d, respectively. It is very clear that both the hoop and axial stresses are much different from each location. It shows the axial and hoop residual stresses on the outer surface in the angles of 0, 90, 180, and 270° , respectively. The maximum compressive axial residual stress predicted by numerical simulation is in the angle of 270° , and the maximum tensile axial stress is in the angle of 90° . The maximum compressive axial stresses in some cases exceeded minimum values of the base metal.

5. CONCLUSION

Numerical simulation of dissimilar P460NH and E355K2 steel weld pipe joint is an effective way to predict the residual stress and hardness test using 2-D and 3-D model. The conclusions drawn from the P460NH and E355K2 weld simulation results are as following:

- Both the 3-D FE model and the 2-D FE model predicted very satisfactory temperature cycles and welding residual stress fields. Using the 3-D model, the detailed information on welding residual stress fields can be captured. The 2-D model cannot provide a minute temperature and residual stresses distributions, but a large amount of computational time can be saved;
- Three-dimensional FE analysis is essential to accurately predict the axial and hoop residual stresses in girth welds of steel pipes which change spatially due to the travelling arc and welding start/stop effects;
- The distribution characteristics of residual stress distribution in the welded pipe can be described as follows. In 2-D modeling, the axial residual stresses on the inside surface and the outside surface have a contrary distribution. In 3-D modeling weld zone and its vicinity in different angles, a tensile axial residual stress is produced on the outside surface, and away from the weld centerline, compressive axial stress is formed. On the outside surface, the shape of hoop stress distribution is very similar to the axial stress. The shape of the hoop stress distribution on the outside surface is very sensitive to the distance from the weld centerline;
- Residual stress has a maximum value at a circumferential angle of 270° in both axial and hoop direction because of changing the weld pool trajectory. On the other hand, the residual stress under tension has a maximum at 90° circumferential angle in both axial and hoop directions.

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