# Acoustic Determination of Thermophysical Properties and Critical Parameters for R410A and Critical Line of $x \mathrm{CO}_{2}+(1-x)$ R410A 

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#### Abstract

The thermophysical properties and critical parameters for the alternative refrigerant R410A (a near azeotropic refrigerant mixture containing difluoromethane (R32) and pentafluoroethane (R125) ( $50 \mathrm{wt} \%$ of each)) were investigated using two different acoustic techniques. The critical behavior of the system $x \mathrm{CO}_{2}+(1-x) \mathrm{R} 410 \mathrm{~A}$ was also investigated. Experimental data of speed of sound in liquid R410A from ( 253 to 338 ) K and pressures up to 65 MPa were measured using a pulse-echo method. Derived thermodynamic properties were calculated, combining our experimental data with density and isobaric heat capacity values published by other authors. Measurements of the critical temperature $T_{\mathrm{c}}$ and pressure $p_{\mathrm{c}}$ on (R410A) and mixtures of $x \mathrm{CO}_{2}+(1-x) \mathrm{R} 410 \mathrm{~A}$ were performed using another simple ultrasonic time-delay technique. The binary critical line was determined over the whole composition range, showing that this system deviates only slightly from ideality since the critical line is a continuous line. The Peng-Robinson equation of state with conventional mixing and combining rules was used to correlate the binary experimental data.


## Introduction

Fluorinated alkanes (HFCs) represent the vast majority of chlorofluorocarbon (CFCs) and hydrochlorofluorocarbon (HCFC) substitutes. The technical challenges of the CFC phase-out are mostly overcome, but the HCFC phase-out cannot be regarded as finalized.

R410A, is a near azeotropic refrigerant mixture containing $50 \mathrm{wt} \%$ of pentafluoroethane (R125) $+50 \mathrm{wt} \%$ of difluoromethane (R32), that can be used as a substitute for R22 and also R13B1. One of the characteristics of the refrigerant R410A is its high vapor pressure, which, when applied to a heat pump, has led to significantly improved refrigerating capacity as compared to R22, R407C, and even propane. Results show that the use of R410A in optimized systems will increase the coefficient of performance when compared to R22, allowing the reduction of the overall costs when compared to other fluids. Also, the environmentally friendly R410A enjoys an important advantage because of the reduction in equipment size. ${ }^{1}$ In other words, it is a high-pressure/high-efficiency refrigerant of the future, which in the long term is considered to replace R22. ${ }^{2-4}$ The R32/125 system is perhaps the most widely studied system of all mixtures that have ever been measured, even compared to the well-measured systems methane/ethane, nitrogen/argon, and dry air. The data span the entire composition range and were measured at temperatures and pressures that cover nearly the entire range of practical fluid states. ${ }^{5,6}$ As to environmental aspects, it has zero ozone depletion potential and a halocarbon global warming potential similar to R22. Despite being considered one of the most suitable HFC replacements for R22, there are few property measurements of the liquid phase reported, namely, the speed of sound. ${ }^{5}$

[^0]The speed of sound $(u)$ is a thermodynamic property that can be experimentally determined with great accuracy over a large range of temperature and pressure conditions, both in the liquid and in the gas phases. Since $u$ can be related to the first pressure partial derivative of the density, high accuracy speed of sound data can be used to enhance the development of equations of state (EOS). Also, it is very useful as a source of information to compute the values of other thermodynamic properties difficult to obtain at extreme experimental conditions, such as calorimetric data at high pressures. In the present work, the ultrasonic speed of propagation in liquid R410A has been measured with an apparatus designed to operate at high pressures. This apparatus has been tested and the results reported for other alternative refrigerants. ${ }^{7-11}$ The experimental data were fitted and used together with density and isobaric heat capacity data from REFPROP ${ }^{12}$ to calculate, through an integration method, ${ }^{7,13-15}$ several thermodynamic properties such as the isentropic and isothermal compressibilities ( $\kappa_{\mathrm{S}}$ and $\kappa_{\mathrm{T}}$ ), the isobaric thermal expansion coefficient $\left(\alpha_{p}\right)$, the thermal pressure coefficient $\left(\gamma_{\mathrm{V}}\right)$, the isenthalpic Joule-Thomson coefficient $\left(\mu_{\mathrm{JT}}\right)$, the isobaric and isochoric heat capacities $\left(C_{\mathrm{p}}\right.$ and $\left.C_{\mathrm{V}}\right)$, and enthalpy and entropy variations relative to a reference state.
Recent works have shown that it seems likely that blends of new refrigerants rather than pure substances will replace the common CFCs. Vapor-liquid equilibrium data are required in order to evaluate the performance of refrigeration cycles. Moreover, apart from other important physicochemical properties, the knowledge of critical parameters is essential to design a simple process. Mixtures of partially fluorinated gases with $\mathrm{CO}_{2}$ can be used as refrigerants or as modifiers to improve solubility in supercritical fluids. Therefore, knowledge of their phase behavior is crucial. In this work, critical properties of the refrigerant R 410 A and their mixtures with $\mathrm{CO}_{2}$ were investigated. To that end, we have applied an acoustic simple technique that proved to be a reliable method to explore the

Table 1. Experimental Speed of Sound, $\boldsymbol{u} / \mathrm{m} \cdot \mathrm{s}^{-1}$, for Alternative Refrigerant Mixture R410A as a Function of Temperature $\boldsymbol{T}$ and Pressure $\boldsymbol{p}$

| $T=253.09 \mathrm{~K}$ |  | $T=258.09 \mathrm{~K}$ |  | $T=265.61 \mathrm{~K}$ |  | $T=273.12 \mathrm{~K}$ |  | $T=280.72 \mathrm{~K}$ |  | $T=288.20 \mathrm{~K}$ |  | $T=298.20 \mathrm{~K}$ |  | $T=308.25 \mathrm{~K}$ |  | $T=318.17 \mathrm{~K}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $p$ | $u$ | $p$ | $u$ | $p$ | $u$ | $p$ | $u$ | $p$ | $u$ | $p$ | $u$ | $p$ | $u$ | $p$ | $u$ | $p$ | $u$ |
| MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ |
| 1.18 | 684.1 | 1.00 | 656.5 | 1.19 | 618.9 | 0.83 | 574.5 | 1.03 | 535.2 | 1.35 | 496.0 | 1.68 | 440.6 | 2.17 | 383.3 | 2.75 | 323.4 |
| 1.26 | 684.8 | 1.12 | 657.6 | 1.34 | 620.0 | 0.83 | 574.5 | 1.06 | 535.5 | 1.42 | 497.2 | 1.68 | 440.2 | 2.17 | 383.4 | 2.76 | 323.6 |
| 1.38 | 685.8 | 1.26 | 658.8 | 1.50 | 621.6 | 1.22 | 578.8 | 1.15 | 536.9 | 148 | 498.0 | 1.68 | 440.3 | 2.25 | 384.9 | 2.88 | 327.2 |
| 1.50 | 686.8 | 1.38 | 659.7 | 1.75 | 623.8 | 1.36 | 580.1 | 1.26 | 538.0 | 1.55 | 499.0 | 1.75 | 441.7 | 2.38 | 387.7 | 3.01 | 330.8 |
| 1.62 | 687.7 | 1.50 | 660.8 | 2.01 | 626.1 | 1.50 | 581.6 | 1.38 | 539.4 | 1.65 | 500.3 | 1.88 | 444.0 | 2.51 | 390.5 | 3.25 | 337.8 |
| 1.75 | 688.8 | 1.50 | 661.0 | 2.25 | 628.3 | 1.75 | 584.2 | 1.50 | 540.9 | 1.75 | 501.7 | 2.01 | 446.0 | 2.75 | 395.6 | 3.50 | 344.5 |
| 2.01 | 690.6 | 1.63 | 662.2 | 2.51 | 630.7 | 2.01 | 586.8 | 1.63 | 542.3 | 1.88 | 503.4 | 2.13 | 448.2 | 3.01 | 400.6 | 4.00 | 356.6 |
| 2.25 | 692.6 | 1.75 | 663.1 | 2.75 | 633.0 | 2.25 | 589.4 | 1.75 | 543.8 | 2.01 | 505.1 | 2.25 | 450.0 | 3.50 | 409.9 | 4.50 | 367.9 |
| 2.51 | 694.3 | 2.01 | 665.1 | 3.01 | 635.2 | 2.51 | 591.9 | 2.01 | 546.7 | 2.25 | 508.2 | 2.51 | 454.0 | 4.00 | 418.9 | 5.00 | 378.2 |
| 2.75 | 696.4 | 2.25 | 667.1 | 3.50 | 639.4 | 2.75 | 594.1 | 2.25 | 549.6 | 2.51 | 511.3 | 2.75 | 457.8 | 4.50 | 427.4 | 6.01 | 397.4 |
| 3.01 | 698.4 | 2.51 | 669.2 | 4.00 | 644.0 | 3.01 | 596.9 | 2.51 | 552.3 | 2.75 | 514.7 | 3.01 | 461.7 | 5.00 | 435.6 | 7.01 | 414.5 |
| 3.50 | 702.3 | 2.75 | 671.4 | 4.50 | 648.1 | 3.50 | 601.6 | 2.75 | 555.0 | 3.01 | 517.7 | 3.50 | 469.1 | 6.01 | 450.5 | 8.00 | 429.7 |
| 4.00 | 705.0 | 3.01 | 673.4 | 5.00 | 652.1 | 4.00 | 606.4 | 3.01 | 557.9 | 3.50 | 524.0 | 4.00 | 475.9 | 7.01 | 464.7 | 9.00 | 444.1 |
| 4.50 | 709.5 | 3.50 | 677.1 | 6.01 | 660.6 | 4.50 | 611.0 | 3.50 | 563.2 | 4.00 | 529.7 | 4.50 | 483.0 | 8.00 | 477.8 | 10.00 | 457.3 |
| 5.00 | 713.3 | 4.00 | 681.1 | 7.01 | 668.4 | 5.00 | 615.4 | 4.00 | 568.5 | 4.50 | 535.6 | 5.00 | 489.9 | 9.00 | 490.2 | 12.00 | 481.7 |
| 6.01 | 720.6 | 4.50 | 685.3 | 8.00 | 676.1 | 6.01 | 624.4 | 4.50 | 573.7 | 5.00 | 541.1 | 6.01 | 502.6 | 10.00 | 501.7 | 14.00 | 503.8 |
| 7.01 | 727.4 | 5.00 | 689.1 | 9.00 | 683.8 | 7.01 | 633.3 | 5.00 | 579.0 | 6.01 | 552.0 | 7.01 | 514.4 | 12.00 | 523.6 | 16.01 | 524.3 |
| 8.00 | 734.2 | 6.01 | 696.6 | 10.00 | 691.0 | 8.00 | 641.5 | 6.01 | 588.6 | 7.01 | 562.3 | 8.00 | 525.7 | 14.00 | 543.5 | 18.00 | 543.1 |
| 9.00 | 740.8 | 7.01 | 703.9 | 12.00 | 705.2 | 9.00 | 649.3 | 7.01 | 598.3 | 8.00 | 572.3 | 9.00 | 536.4 | 16.01 | 562.1 | 20.00 | 560.6 |
| 10.00 | 747.5 | 8.00 | 711.0 | 14.00 | 718.9 | 10.00 | 657.6 | 8.00 | 607.1 | 9.00 | 582.0 | 10.00 | 546.7 | 18.00 | 579.5 | 23.00 | 585.3 |
| 12.00 | 760.2 | 9.00 | 718.1 | 16.01 | 732.1 | 12.00 | 672.7 | 9.00 | 615.9 | 10.00 | 591.2 | 12.00 | 566.4 | 20.00 | 596.0 | 26.00 | 607.9 |
| 14.00 | 772.5 | 10.00 | 724.8 | 18.00 | 744.4 | 14.00 | 687.3 | 10.00 | 624.5 | 12.00 | 608.7 | 14.00 | 584.5 | 23.00 | 619.0 | 29.01 | 628.9 |
| 16.01 | 784.5 | 12.00 | 738.2 | 20.00 | 756.2 | 16.01 | 701.1 | 12.00 | 640.6 | 14.00 | 625.1 | 16.01 | 601.6 | 26.00 | 640.0 | 32.00 | 648.5 |
| 18.00 | 795.3 | 14.00 | 751.2 | 23.00 | 773.6 | 18.00 | 714.5 | 14.00 | 656.1 | 16.01 | 640.6 | 16.01 | 601.3 | 29.01 | 660.1 | 35.00 | 667.3 |
| 20.00 | 806.9 | 16.01 | 763.4 | 26.00 | 790.1 | 20.00 | 727.0 | 16.01 | 670.9 | 18.00 | 655.7 | 18.00 | 617.3 | 32.00 | 678.9 | 40.00 | 696.7 |
| 23.00 | 822.9 | 18.00 | 775.0 | 29.01 | 806.2 | 23.00 | 745.4 | 18.00 | 684.9 | 20.00 | 669.7 | 20.00 | 632.4 | 35.00 | 697.2 | 45.00 | 723.8 |
| 26.00 | 837.9 | 20.00 | 786.7 | 32.00 | 821.5 | 26.00 | 762.7 | 20.00 | 698.3 | 23.00 | 689.6 | 23.00 | 653.8 | 40.00 | 725.2 | 50.00 | 749.1 |
| 29.01 | 852.8 | 23.00 | 803.2 | 35.00 | 835.5 | 29.01 | 779.0 | 23.00 | 717.0 | 26.00 | 708.7 | 26.00 | 673.7 | 45.00 | 751.3 | 55.00 | 772.8 |
| 32.00 | 867.2 | 26.00 | 818.7 | 40.00 | 858.8 | 32.00 | 794.8 | 26.00 | 735.5 | 29.01 | 727.0 | 29.01 | 693.0 | 50.00 | 775.7 | 60.00 | 795.7 |
| 35.00 | 880.5 | 29.01 | 833.7 | 45.00 | 880.8 | 35.00 | 809.6 | 29.01 | 752.6 | 32.00 | 744.0 | 32.00 | 711.1 | 55.00 | 798.8 | 65.00 | 817.3 |
| 40.00 | 902.6 | 32.00 | 848.7 | 50.00 | 902.0 | 40.00 | 833.5 | 32.00 | 769.2 | 35.00 | 760.4 | 35.00 | 728.5 | 60.00 | 821.0 |  |  |
| 45.00 | 923.6 | 35.00 | 862.6 | 55.00 | 922.4 | 45.00 | 856.4 | 35.00 | 785.1 | 40.00 | 786.1 | 40.00 | 754.9 | 65.00 | 841.9 |  |  |
| 50.00 | 943.8 | 40.00 | 885.0 | 60.00 | 941.6 | 50.00 | 877.8 | 40.00 | 809.7 | 45.00 | 810.0 | 45.00 | 780.3 |  |  |  |  |
| 55.00 | 962.8 | 45.00 | 906.3 | 65.00 | 959.9 | 55.00 | 898.7 | 45.00 | 833.1 | 50.00 | 833.0 | 50.00 | 803.7 |  |  |  |  |
| 60.00 | 981.2 | 50.00 | 926.7 |  |  | 60.00 | 919.0 | 50.00 | 855.3 | 55.00 | 854.6 | 55.00 | 826.3 |  |  |  |  |
| 65.00 | 999.5 | 55.00 | 946.6 |  |  | 65.00 | 938.0 | 55.00 | 876.4 | 60.00 | 875.4 | 60.00 | 847.6 |  |  |  |  |
|  |  | 60.00 | 965.6 |  |  |  |  | 60.00 | 898.9 | 65.00 | 895.5 | 65.00 | 868.2 |  |  |  |  |
|  |  | 65.00 | 983.3 |  |  |  |  | 65.00 | 916.7 |  |  |  |  |  |  |  |  |


| $T=328.17 \mathrm{~K}$ |  |  |  | $T=338.16 \mathrm{~K}$ |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\frac{p}{\mathrm{MPa}}$ | $\frac{u}{\mathrm{~m} \cdot \mathrm{~s}^{-1}}$ | $\frac{p}{\mathrm{MPa}}$ | $\frac{u}{\mathrm{~m} \cdot \mathrm{~s}^{-1}}$ | $\frac{p}{\mathrm{MPa}}$ | $\frac{u}{\mathrm{~m} \cdot \mathrm{~s}^{-1}}$ | $\frac{p}{\mathrm{MPa}}$ | $\frac{u}{\mathrm{~m} \cdot \mathrm{~s}^{-1}}$ |
| 3.47 | 257.9 | 14.00 | 464.8 | 4.34 | 183.3 | 18.00 | 472.9 |
| 3.47 | 258.1 | 16.01 | 487.3 | 4.41 | 189.7 | 20.00 | 493.2 |
| 3.47 | 258.0 | 18.00 | 507.4 | 4.50 | 200.4 | 23.00 | 521.1 |
| 3.59 | 263.4 | 20.00 | 526.4 | 4.75 | 217.6 | 26.00 | 546.3 |
| 3.72 | 269.8 | 23.00 | 552.3 | 5.00 | 231.3 | 29.01 | 569.7 |
| 3.86 | 275.4 | 26.00 | 576.7 | 5.50 | 255.4 | 32.00 | 591.3 |
| 4.00 | 281.3 | 29.01 | 598.7 | 6.01 | 273.8 | 35.00 | 611.3 |
| 4.25 | 290.2 | 32.00 | 619.2 | 7.01 | 303.5 | 40.00 | 642.8 |
| 4.50 | 298.5 | 35.00 | 638.6 | 8.00 | 327.5 | 45.00 | 671.7 |
| 5.00 | 313.4 | 40.00 | 669.0 | 9.00 | 348.4 | 50.00 | 698.8 |
| 6.01 | 339.3 | 45.00 | 697.4 | 10.00 | 366.8 | 55.00 | 724.1 |
| 7.01 | 360.6 | 50.00 | 723.3 | 12.00 | 398.9 | 60.00 | 747.7 |
| 8.00 | 379.4 | 55.00 | 747.8 | 14.00 | 426.3 | 65.00 | 770.1 |
| 9.00 | 396.7 | 60.00 | 770.9 | 16.01 | 450.8 |  |  |
| 10.00 | 412.2 | 65.00 | 793.1 |  |  |  |  |
| 12.00 | 440.2 |  |  |  |  |  |  |

critical behavior of multicomponent systems as reported in previous works. ${ }^{11,16-23}$ Measurements of the critical temperature $T_{\mathrm{c}}$ and pressure $p_{\mathrm{c}}$ of the $\mathrm{CO}_{2}+\mathrm{R} 410 \mathrm{~A}$ mixtures were performed over the whole composition range. The Peng-Robinson equation of state (PR EOS), with conventional mixing and combining rules, was used to correlate the experimental data.

## Experimental Section

Materials. The R410A used in the experimental work was supplied by Solvay Fluor und Derivate GmbH, with a stated composition of $50 \mathrm{wt} \%$ of pentafluoroethane (R125) +50 wt
\% of difluoromethane (R32). $\mathrm{CO}_{2}$ was supplied by Air Liquide and had a stated purity greater than $99.99 \%$. The mixtures were prepared in small stainless steel cylinders, and the compositions were determined by weighing.

Speed of Sound Measurements. The speed of sound of liquid R410A was measured using an apparatus designed for high pressures, with an acoustic cell placed inside a stainless steel container, using a pulse-echo method at a frequency of 1 MHz as previously described. ${ }^{7-11}$ The cell calibration was performed by measuring the speed of sound of pure $\mathrm{CCl}_{4}$ at 10 MPa and 298.15 K and comparing the obtained value with data reported
in the literature. ${ }^{24,25}$ To cover the complete experimental temperature and pressure ranges, the calibration was expanded by the use of the thermal and pressure behavior of the pure copper spacers (purity better than 99.99 \%). The steel container and the acoustic cell were thermostated inside a Dewar vessel within the temperature range from ( 253 to 388 ) K, with a control system based on two-stage cascaded thermostatic baths. The long-term stability of the complete system was within $\pm 0.02$ K in the whole range, and the temperature measurements were taken with a four-wire platinum resistance thermometer previously calibrated to an uncertainty of $\pm 0.01 \mathrm{~K}$ on ITS 90 scale. Pressure measurements were performed in two separate ranges with Setra Systems Inc. pressure transducers, one from vapor pressure to 35 MPa and the other up to 65 MPa . The pressure sensors were calibrated against a dead weight gauge with an uncertainty of $\pm 0.025 \mathrm{MPa}$ in the lower range and $\pm 0.076$ MPa in the higher range.

Taken into account all the error sources, the accuracy of the speed of sound measurements is estimated to be better than $\pm$ $0.3 \mathrm{~m} \cdot \mathrm{~s}^{-1}$. Several measurements were repeated at the same pressure and temperature conditions, allowing us to estimate a precision of $\pm 0.1 \mathrm{~m} \cdot \mathrm{~s}^{-1}$.

Measurements of Critical Data. The experimental apparatus for the critical data acquisition has already been described in detail ${ }^{18,19}$ and was used unmodified in this study.

The acoustic cell is constructed from a stainless steel $3 / 8 \mathrm{in}$. cross piece, with its inner diameter bored out to provide an acoustic cavity of about 5 mL volume. Pressure was generated via a hand pump (High-Pressure Equipment Co., model 62-610) and monitored with a pressure transducer (Omega, model PX931) with a precision of $\pm 0.01 \mathrm{MPa}$. Temperature was measured on the ITS-90 with a precision of $\pm 0.001 \mathrm{~K}$ with a calibrated platinum resistance thermometer (Tinsley) connected to a digital multimeter (Keithley, model 2000). A pulse generator (Wavetek, model 80) provided the acoustic signal, with a frequency of about 300 kHz . The signal was fed into the acoustic cavity via a piezoelectric ceramic transducer disk (Morgan Matroc Ltd., PC4D) with a thickness of 2 mm and a diameter of 6 mm . A second, identical transducer monitored the signal at the other end of the cavity. The transit times of the pulse across the acoustic cavity were displayed in an oscilloscope (Tektronix, model TDS-340). The acoustic cell was immersed in a 40 L water bath, thermostated with a refrigeration unit and a temperature controller from Hart Scientific, with a RTD probe. This unit provides a temperature control within $\pm$ 0.004 K.

A mixture with a desired composition was obtained by weighing and was pumped into the acoustic cell. The pressure was increased with the hand pump until only minute changes in the acoustic signal were observed, meaning that a liquid like state with a low compressibility was present in the acoustic cell. The system was allowed to equilibrate for at least 30 min at the desired starting temperature. Experiments were carried out at constant temperature, and pressure was lowered until a maximum in time delay was observed. Temperature was then systematically varied, and the procedure was repeated. The temperature and pressure values corresponding to the absolute maximum of time delay in the ensemble of the isothermal curves $\left(t_{\text {delay }} / \mu \mathrm{s}\right.$ vs $\left.p / \mathrm{MPa}\right)$ were taken as the critical temperature and critical pressure.

## Results and Discussion

Speed of Sound of Liquid R410A. The apparatus was used to take a total of 376 experimental speed of sound measurements


Figure 1. Experimental speed of sound, $u$, for R410A as a function of pressure at:,$+ 253.09 \mathrm{~K} ;-258.09 \mathrm{~K} ; *, 265.61 \mathrm{~K} ; \bigcirc, 273.12 \mathrm{~K} ; \Delta$, $280.72 \mathrm{~K} ; \diamond, 288.20 \mathrm{~K} ; \square, 298.20 \mathrm{~K} ; \bullet, 308.25 \mathrm{~K} ; \Delta, 318.17 \mathrm{~K} ; \star, 328.17$ $\mathrm{K} ; \boldsymbol{\square}, 338.16 \mathrm{~K} ;-$, saturation line; -- , critical pressure.

Table 2. Orthobaric Speed of Sound Data for R410A

| $T$ | $p_{\sigma}$ | $u_{\sigma}$ | $T$ | $p_{\sigma}$ | $u_{\sigma}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| K | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ | K | MPa | $\mathrm{m} \cdot \mathrm{s}^{-1}$ |
| 253.09 | 0.40 | 677.8* | 298.20 | 1.65 | 440.4 |
| 258.09 | 0.48 | 652.1* | 308.25 | 2.14 | 383.4 |
| 265.61 | 0.62 | 613.2* | 318.17 | 2.72 | 323.4 |
| 273.12 | 0.80 | 574.5 | 328.17 | 3.43 | 256.3* |
| 280.72 | 1.01 | 535.2 | 338.16 | 4.28 | 177.4* |
| 288.20 | 1.26 | 494.9* |  |  |  |

${ }^{a}$ An asterisk (*) indicates an extrapolated value.
of the alternative refrigerant mixture R410A, within the temperature range from ( 253 to 338 ) K and pressure range up to 65 MPa . These data are shown in Table 1. The measurements were performed and organized as isotherms, within a total of 11, and are plotted in Figure 1. The consistency of the individual isotherms was tested by fitting each one with

$$
\begin{equation*}
u=\sum_{i=0}^{2} A_{i}\left[\ln \left(p-B_{i}\right)\right]^{i} \tag{1}
\end{equation*}
$$

The standard deviations of the fits are smaller than $0.18 \mathrm{~m} \cdot \mathrm{~s}^{-1}$ for isotherms in which the reduced temperature is lower than 0.93 . For the isotherms at ( 328 and 338) K, which correspond to reduced temperatures of 0.95 and 0.98 , the standard deviations of the fits are $0.32 \mathrm{~m} \cdot \mathrm{~s}^{-1}$ and $0.67 \mathrm{~m} \cdot \mathrm{~s}^{-1}$, respectively. This increase in the standard deviation of the fits can be explained by the proximity of the critical temperature and consequently abrupt changes in the speed of sound measures when we are close to the critical point, as can be seen in Figure 1.

It was possible to measure the saturated liquid (orthobaric) speed of sound, $u_{\sigma}$, for the isotherms between 273 K and 318 K except for 288 K . In the cases where the direct measurement was not possible, eq 1 was used to extrapolate the speed of sound at this temperature to the vapor pressure (REFPROP). ${ }^{12}$ The orthobaric speed of sound data presented in Table 2 were fitted to

$$
\begin{equation*}
u_{\sigma}=\sum_{i=0}^{4} a_{i}\left[\ln \left(\frac{T}{T_{\mathrm{c}}}\right)\right]^{i} \tag{2}
\end{equation*}
$$

(critical data $T_{\mathrm{c}}=344.90 \mathrm{~K}$, measured value in this work) with the coefficients of Table 3. The standard deviation of the fit was $0.70 \mathrm{~m} \cdot \mathrm{~s}^{-1}$.

Table 3. Coefficients of Equation 2 for the Orthobaric Speed of Sound, $\boldsymbol{u}_{\boldsymbol{\sigma}}$

| $i$ | $a_{i}$ | $i$ | $a_{i}$ |
| :---: | ---: | :---: | :---: |
| 0 | $1.182664 \mathrm{E}+02$ | 3 | $-2.735656 \mathrm{E}+04$ |
| 1 | $-3.183315 \mathrm{E}+03$ | 4 | $-3.057635 \mathrm{E}+04$ |
| 2 | $-9.987818 \mathrm{E}+03$ |  |  |

Table 4. Coefficients of Equation 3 for the Speed of Sound of R410A, Valid from $T=$ (253 to 338) $K$ and Pressures from Saturation up to 65 MPa

| $a_{i j}$ | $j$ |  |  |
| :---: | ---: | ---: | ---: |
|  | 0 | 1 | 2 |
|  | $1.7954580 \mathrm{E}+03$ | $-1.0022012 \mathrm{E}+01$ | $1.3836079 \mathrm{E}-02$ |
| 1 | $-6.2369087 \mathrm{E}+01$ | $4.6733643 \mathrm{E}-01$ | $-8.0659148 \mathrm{E}-04$ |
| 2 | $1.9859669 \mathrm{E}-01$ | $-3.0301947 \mathrm{E}-03$ | $8.9030601 \mathrm{E}-06$ |
|  |  | $l$ |  |
| $b_{k l}$ |  | 1 | 2 |
| $k$ | 0 | $-3.3868948 \mathrm{E}-03$ | $1.1342597 \mathrm{E}-06$ |
| 0 | 1 |  | $2.6896918 \mathrm{E}-04$ |
| 1 | $-4.5482056 \mathrm{E}-02$ | $-2.9582158 \mathrm{E}-07$ |  |
| 2 | $4.5276779 \mathrm{E}-04$ | $-4.4360736 \mathrm{E}-06$ | $1.0375652 \mathrm{E}-08$ |

Finally, the set of experimental data excluding the isotherm at 338 K , was fitted to

$$
\begin{equation*}
u=\frac{\sum_{i=0}^{2} \sum_{j=0}^{2} a_{i j} p^{i} T^{j}}{\sum_{k=0}^{2} \sum_{l=0}^{2} b_{k l} p^{k} T} \tag{3}
\end{equation*}
$$

with the coefficients shown in Table 4 and a standard deviation of $0.40 \mathrm{~m} \cdot \mathrm{~s}^{-1}$. The comparison between our fitted speed of sound data and the results obtained from REFPROP ${ }^{12}$ are presented in Figure 2. The deviations are slightly high, but this can be related with the mixing rule used in the commercial package. As far as we are aware of there are no other speed of sound experimental data to which the present work can be compared.

Derived Thermodynamic Properties. The speed of sound, $u$, is directly related to the pressure derivative of the density, $\rho$, through eq 4 , in which the subscript $S$ denotes the condition of constant entropy:

$$
\begin{equation*}
\left(\frac{\partial \rho}{\partial p}\right)_{\mathrm{S}}=\frac{1}{u^{2}} \tag{4}
\end{equation*}
$$

This derivative is related to the isothermal pressure derivative


Figure 2. Percentage deviations $\left\{100 \cdot\left(u_{\mathrm{fit}}-u_{\mathrm{ref}} / u_{\mathrm{ref}}\right\}\right.$ of the calculated speed of sound, $u_{\text {fit }}$, to REFPROP ${ }^{12}$ data, $u_{\text {ref. }}$. of R410A: $+260 \mathrm{~K} ; *, 270$ K; $\Delta, 280 \mathrm{~K} ; \square, 290 \mathrm{~K} ; \diamond, 295 \mathrm{~K} ; \mathbf{\Delta}, 300 \mathrm{~K} ; ~ ■, 310 \mathrm{~K} ; \downarrow, 320 \mathrm{~K}$.

Table 5. Coefficients Used To Fit the Literature Density, $\rho_{i}$, and Heat Capacity Data, $C_{j}$, with Equationss 14 and 15, Respectively

| $i$ | $d_{i}$ | $c_{i}$ |
| :---: | ---: | ---: |
| 0 | $6.635945 \mathrm{E}+04$ | $-1.260861 \mathrm{E}+07$ |
| 1 | $-1.191944 \mathrm{E}+03$ | $3.285457 \mathrm{E}+05$ |
| 2 | $8.778011 \mathrm{E}+00$ | $-3.662179 \mathrm{E}+03$ |
| 3 | $-3.237582 \mathrm{E}-02$ | $2.263942 \mathrm{E}+01$ |
| 4 | $5.969828 \mathrm{E}-05$ | $-8.383379 \mathrm{E}-02$ |
| 5 | $-4.408465 \mathrm{E}-08$ | $1.859637 \mathrm{E}-04$ |
| 6 |  | $-2.288239 \mathrm{E}-07$ |
| 7 |  | $1.204971 \mathrm{E}-10$ |

and the isobaric temperature derivative of the density through eq 5 , where $C_{\mathrm{p}}$ is the specific heat capacity at constant pressure:

$$
\begin{equation*}
\left(\frac{\partial \rho}{\partial p}\right)_{\mathrm{S}}=\left(\frac{\partial \rho}{\partial p}\right)_{\mathrm{T}}-\frac{\mathrm{T}}{\rho^{2} \cdot C_{\mathrm{p}}}\left(\frac{\partial \rho}{\partial T}\right)_{\mathrm{p}}^{2} \tag{5}
\end{equation*}
$$

Rearranging the last equation and combining it with eq 4 , follows eq 6, which also incorporates the definition of the thermal expansion coefficient, $\alpha_{p}$ :

$$
\begin{gather*}
\left(\frac{\partial \rho}{\partial p}\right)_{\mathrm{T}}=\frac{1}{u^{2}}+\frac{T}{C_{\mathrm{p}}} \cdot \alpha_{\mathrm{p}}^{2}  \tag{6}\\
\alpha_{\mathrm{p}} \tag{7}
\end{gather*}=-\frac{1}{\rho}\left(\frac{\partial \rho}{\partial T}\right)_{\mathrm{p}} .
$$

It can also be shown that the pressure partial derivative of the isobaric heat capacity can calculated with eq 8 :

$$
\begin{equation*}
\left(\frac{\partial C_{\mathrm{p}}}{\partial p}\right)_{\mathrm{T}}=-\frac{T}{\rho}\left[\alpha_{\mathrm{p}}^{2}+\left(\frac{\partial \alpha_{\mathrm{p}}}{\partial T}\right)_{\mathrm{p}}\right] \tag{8}
\end{equation*}
$$

This way, given an isobar of the density and of $C_{\mathrm{p}}$, it is possible to integrate eqs 6 and 8 over the pressure thus obtaining the $(p, \rho, T)$ and ( $p, C_{\mathrm{p}}, T$ ) surfaces within the range of pressure and temperature of the experimental speed of sound data. The numerical integration procedure also allows, through the use of eqs 9 to 13 , the calculation of other properties, such as the isentropic compressibility $\kappa_{\mathrm{S}}$; the isothermal compressibility $\kappa_{\mathrm{T}}$; the isochoric heat capacity $C_{\mathrm{v}}$; the thermal pressure coefficient $\gamma_{\mathrm{v}}$; and the isenthalpic Joule-Thomson coefficient $\mu_{\mathrm{JT}}$ :

$$
\begin{gather*}
\kappa_{\mathrm{S}}=\frac{1}{\rho \cdot u^{2}}  \tag{9}\\
\kappa_{\mathrm{T}}=\frac{1}{\rho}\left(\frac{\partial \rho}{\partial \mathrm{p}}\right)_{\mathrm{T}}  \tag{10}\\
C_{\mathrm{v}}=\frac{C_{\mathrm{p}}}{\gamma}  \tag{11}\\
\gamma_{\mathrm{v}}=\frac{\alpha_{\mathrm{p}}}{\kappa_{\mathrm{T}}}  \tag{12}\\
\mu_{\mathrm{JT}}=\frac{T \cdot \alpha_{\mathrm{p}}-1}{\rho \cdot C_{\mathrm{p}}} \tag{13}
\end{gather*}
$$

In the present work, because the available experimental density is scarce and no isobaric heat capacity data exist, the authors used data from the commercial available program REFPROP ${ }^{12}$ at a pressure of 5 MPa and temperatures from (240

Table 6. Calculated Densities $\rho$ for $\mathbf{R 4 1 0 A}$ as a Function of Temperature $T$ and Pressure $p$

| $p / \mathrm{MPa}$ | $\rho /\left(\mathrm{kg} \cdot \mathrm{m}^{-3}\right)$ |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $T=260 \mathrm{~K}$ | $T=270 \mathrm{~K}$ | $T=280 \mathrm{~K}$ | $T=290 \mathrm{~K}$ | $T=295 \mathrm{~K}$ | $T=300 \mathrm{~K}$ | $T=310 \mathrm{~K}$ | $T=320 \mathrm{~K}$ |
| 1.0 | 1223.1 | 1184.6 | 1142.8 |  |  |  |  |  |
| 3.0 | 1231.0 | 1194.1 | 1154.4 | 1111.5 | 1088.4 | 1063.8 | 1008.2 | 937.6 |
| 5.0 | 1238.4 | 1202.8 | 1165.0 | 1124.5 | 1103.1 | 1080.5 | 1031.2 | 973.2 |
| 10.0 | 1255.1 | 1222.5 | 1188.1 | 1152.1 | 1133.3 | 1114.1 | 1073.6 | 1029.7 |
| 15.0 | 1270.1 | 1239.6 | 1207.8 | 1174.8 | 1157.9 | 1140.6 | 1105.0 | 1067.8 |
| 20.0 | 1283.5 | 1254.9 | 1225.2 | 1194.4 | 1178.7 | 1162.9 | 1130.5 | 1097.3 |
| 25.0 | 1295.9 | 1268.7 | 1240.7 | 1211.8 | 1197.1 | 1182.2 | 1152.2 | 1121.7 |
| 30.0 | 1307.3 | 1281.4 | 1254.8 | 1227.4 | 1213.5 | 1199.4 | 1171.1 | 1142.7 |
| 35.0 | 1317.9 | 1293.1 | 1267.7 | 1241.6 | 1228.3 | 1215.0 | 1188.1 | 1161.2 |
| 40.0 | 1327.9 | 1304.1 | 1279.7 | 1254.7 | 1242.0 | 1229.2 | 1203.5 | 1177.9 |
| 45.0 | 1337.3 | 1314.3 | 1290.8 | 1266.8 | 1254.6 | 1242.4 | 1217.7 | 1193.1 |
| 50.0 | 1346.2 | 1323.9 | 1301.3 | 1278.2 | 1266.4 | 1254.6 | 1230.8 | 1207.1 |
| 55.0 | 1354.7 | 1333.0 | 1311.1 | 1288.8 | 1277.5 | 1266.1 | 1243.0 | 1220.1 |
| 60.0 | 1362.8 | 1341.7 | 1320.5 | 1298.9 | 1287.9 | 1276.8 | 1254.5 | 1232.3 |
| 65.0 | 1370.6 | 1349.9 | 1329.3 | 1308.4 | 1297.8 | 1287.0 | 1265.4 | 1243.7 |

Table 7. Calculated Isobaric Heat Capacity $C_{\mathrm{p}}$ for R410A as a Function of Temperature $T$ and Pressure $p$

| $p / \mathrm{MPa}$ | $C_{\mathrm{p}} /\left(\mathrm{J} \cdot \mathrm{kg}^{-1} \cdot \mathrm{~K}^{-1}\right)$ |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $T=260 \mathrm{~K}$ | $T=270 \mathrm{~K}$ | $T=280 \mathrm{~K}$ | $T=290 \mathrm{~K}$ | $T=295 \mathrm{~K}$ | $T=300 \mathrm{~K}$ | $T=310 \mathrm{~K}$ | $T=320 \mathrm{~K}$ |
| 1.0 | 1447 | 1490 | 1543 |  |  |  |  |  |
| 3.0 | 1432 | 1469 | 1514 | 1575 | 1615 | 1664 | 1805 | 2057 |
| 5.0 | 1418 | 1451 | 1490 | 1541 | 1572 | 1608 | 1701 | 1848 |
| 10.0 | 1388 | 1414 | 1446 | 1482 | 1502 | 1523 | 1566 | 1621 |
| 15.0 | 1363 | 1386 | 1413 | 1443 | 1459 | 1473 | 1499 | 1522 |
| 20.0 | 1343 | 1363 | 1387 | 1415 | 1428 | 1440 | 1457 | 1468 |
| 25.0 | 1326 | 1343 | 1366 | 1392 | 1404 | 1416 | 1430 | 1434 |
| 30.0 | 1312 | 1327 | 1348 | 1373 | 1385 | 1397 | 1410 | 1411 |
| 35.0 | 1301 | 1312 | 1332 | 1357 | 1369 | 1381 | 1395 | 1396 |
| 40.0 | 1292 | 1300 | 1317 | 1342 | 1355 | 1367 | 1382 | 1384 |
| 45.0 | 1286 | 1289 | 1305 | 1329 | 1342 | 1355 | 1372 | 1375 |
| 50.0 | 1281 | 1280 | 1293 | 1317 | 1331 | 1344 | 1363 | 1368 |
| 55.0 | 1279 | 1272 | 1282 | 1305 | 1320 | 1333 | 1355 | 1362 |
| 60.0 | 1278 | 1266 | 1273 | 1295 | 1309 | 1324 | 1347 | 1357 |
| 65.0 | 1280 | 1262 | 1265 | 1285 | 1299 | 1314 | 1339 | 1352 |

to 330 ) K . These isobars were fitted to eqs 14 and 15 with the coefficients of Table 5:

$$
\begin{align*}
& \rho(T, 5 \mathrm{MPa})=\sum_{i=0}^{5} d_{i} \cdot T^{i}  \tag{14}\\
& C_{\mathrm{p}}(T, 5 \mathrm{MPa})=\sum_{j=0}^{7} c_{j} \cdot T^{j} \tag{15}
\end{align*}
$$

The integration procedure was implemented both on increasing pressure, up to 65 MPa , and on decreasing pressure down to the saturation line.


Figure 3. Density data of R410A calculated through the integration procedure as a function of pressure: $+260 \mathrm{~K} ; *, 270 \mathrm{~K} ; \Delta, 280 \mathrm{~K}$; $\square$, $290 \mathrm{~K} ; \diamond, 295 \mathrm{~K} ; \mathbf{\Delta}, 300 \mathrm{~K} ; \boldsymbol{\square}, 310 \mathrm{~K} ; \bullet 320 \mathrm{~K}$.

The whole $(p, \rho, T)$ surface obtained from the integration procedure previously described is shown in Figure 3 and Table 6. To test the results, the whole set of calculated density data were fitted with an equation similar to eq 3 , with a standard deviation of $0.008 \%$. Figure 4 shows the percentage deviations between experimental density data ${ }^{26}$ and the density data obtained by integration. From this figure, it is possible to see that all the deviations are roughly within $\pm 0.2 \%$ with the exception of the isotherm at 263.15 K where the deviations are higher. Not shown in Figure 4 there are also density data ${ }^{27}$ that are within $-0.1 \%$ and $+0.3 \%$ from our integration procedure. The complete $\left(p, C_{\mathrm{p}}, T\right)$ surface is plotted in Figure 5 and Table 7. The other calculated thermodynamic


Figure 4. Plot of deviations between our results and the literature experimental densities. Piao et al. $:^{26} \square, 263.15 \mathrm{~K} ; \Delta, 273.15 \mathrm{~K} ; \diamond, 283.15$ $\mathrm{K} ; \boldsymbol{\square}, 293.15 \mathrm{~K} ; \mathbf{\Delta}, 303.15 \mathrm{~K} ; \geqslant 313.15 \mathrm{~K}$.


Figure 5. Isobaric heat capacity, $C_{\mathrm{p}}$, of R410A calculated through the integration procedure:,$+ 260 \mathrm{~K} ; *, 270 \mathrm{~K} ; \Delta, 280 \mathrm{~K} ; \square, 290 \mathrm{~K} ; \diamond, 295 \mathrm{~K}$;
ム, 300 K ; ■, $310 \mathrm{~K} ; \downarrow, 320 \mathrm{~K}$.


Figure 6. Thermal expansion coefficient, $\alpha_{p}$, of R410A calculated through the integration procedure:,$+ 260 \mathrm{~K} ; *, 270 \mathrm{~K} ; \Delta, 280 \mathrm{~K} ; \square, 290 \mathrm{~K} ; \diamond$, 295 K ; $\mathbf{\Delta}, 300 \mathrm{~K}$; 310 K ; $\downarrow, 320 \mathrm{~K}$.
properties are represented in Figures 6 through 8 and Tables 8 to 10 .

Critical Behavior. Our acoustic method was applied to investigate the critical properties of R 410 A and $\mathrm{CO}_{2}+\mathrm{R} 410 \mathrm{~A}$ system. The refrigerant used in this study is a binary azeotropic mixture, which means that its behavior is similar to that one of a pure component. We have considered R410A as a pure component and treated the mixture with $\mathrm{CO}_{2}$ as a binary mixture and not a ternary one. The estimated errors of the critical parameters reported are $\pm 0.1 \mathrm{~K}$ for the critical temperature and 0.05 MPa for the critical pressure. The estimate was based on a comparison of the results of different experiments


Figure 7. Isentropic compressibility, $\kappa_{\mathrm{s}}$, of R410A calculated through the integration procedure:,$+ 260 \mathrm{~K} ; *, 270 \mathrm{~K} ; \Delta, 280 \mathrm{~K} ; \square, 290 \mathrm{~K} ; \diamond, 295 \mathrm{~K}$; ©, 300 K ; 310 K ; $\downarrow, 320 \mathrm{~K}$.


Figure 8. Isothermal compressibility, $\kappa_{\mathrm{T}}$, of R410A calculated through the integration procedure:,$+ 260 \mathrm{~K} ; *, 270 \mathrm{~K} ; \Delta, 280 \mathrm{~K} ; \square, 290 \mathrm{~K} ; \diamond, 295 \mathrm{~K}$; ©, 300 K ; ■, $310 \mathrm{~K} ; \star, 320 \mathrm{~K}$.
performed with the same mixture and is mainly determined by the narrowness of the isotherms studied as during an experiment the pressure decreases in a continuous way.

R410A. The acoustic experiments are carried out at constant temperature, and the pressure is lowered until a maximum in time delay is observed, meaning that it has reached a local maximum in the compressibility, corresponding to a phase transition. The temperature is then systematically varied and the procedure repeated. Several isotherms were measured in the vicinity of the critical temperature. The temperature and pressure values corresponding to the absolute maximum of time delay

Table 8. Calculated Isobaric Thermal Expansion Coefficient $\alpha_{p}$ for R410A as a Function of Temperature $T$ and Pressure $p$

| $p / \mathrm{MPa}$ | $\alpha_{\mathrm{p}} \cdot 10^{3} / \mathrm{K}^{-1}$ |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $T=260 \mathrm{~K}$ | $T=270 \mathrm{~K}$ | $T=280 \mathrm{~K}$ | $T=290 \mathrm{~K}$ | $T=295 \mathrm{~K}$ | $T=300 \mathrm{~K}$ | $T=310 \mathrm{~K}$ | $T=320 \mathrm{~K}$ |
| 1.0 | 3.025 | 3.383 | 3.822 |  |  |  |  |  |
| 3.0 | 2.918 | 3.196 | 3.567 | 4.044 | 4.364 | 4.780 | 6.112 | 8.682 |
| 5.0 | 2.784 | 3.046 | 3.355 | 3.737 | 3.980 | 4.278 | 5.134 | 6.576 |
| 10.0 | 2.534 | 2.741 | 2.962 | 3.209 | 3.350 | 3.509 | 3.908 | 4.476 |
| 15.0 | 2.342 | 2.510 | 2.683 | 2.862 | 2.956 | 3.057 | 3.287 | 3.583 |
| 20.0 | 2.187 | 2.327 | 2.469 | 2.610 | 2.680 | 2.750 | 2.898 | 3.071 |
| 25.0 | 2.059 | 2.176 | 2.298 | 2.415 | 2.470 | 2.524 | 2.626 | 2.734 |
| 30.0 | 1.952 | 2.049 | 2.155 | 2.258 | 2.304 | 2.347 | 2.424 | 2.493 |
| 35.0 | 1.861 | 1.941 | 2.035 | 2.126 | 2.167 | 2.204 | 2.265 | 2.312 |
| 40.0 | 1.783 | 1.847 | 1.930 | 2.014 | 2.051 | 2.084 | 2.136 | 2.169 |
| 45.0 | 1.717 | 1.765 | 1.838 | 1.916 | 1.951 | 1.982 | 2.027 | 2.052 |
| 50.0 | 1.660 | 1.693 | 1.757 | 1.829 | 1.862 | 1.892 | 1.934 | 1.955 |
| 55.0 | 1.611 | 1.629 | 1.684 | 1.751 | 1.783 | 1.811 | 1.853 | 1.871 |
| 60.0 | 1.570 | 1.573 | 1.618 | 1.681 | 1.711 | 1.739 | 1.780 | 1.798 |
| 65.0 | 1.535 | 1.522 | 1.559 | 1.616 | 1.646 | 1.673 | 1.715 | 1.732 |

Table 9. Calculated Isentropic Compressibility $\kappa_{\mathrm{S}}$ for R410A as a Function of Temperature $T$ and Pressure $p$

| $p / \mathrm{MPa}$ | $\kappa_{\mathrm{S}} / \mathrm{GPa}^{-1}$ |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $T=260 \mathrm{~K}$ | $T=270 \mathrm{~K}$ | $T=280 \mathrm{~K}$ | $T=290 \mathrm{~K}$ | $T=295 \mathrm{~K}$ | $T=300 \mathrm{~K}$ | $T=310 \mathrm{~K}$ | $T=320 \mathrm{~K}$ |
| 1.0 | 1.955 | 2.394 | 3.014 |  |  |  |  |  |
| 3.0 | 1.844 | 2.228 | 2.749 | 3.491 | 3.991 | 4.621 | 6.555 | 10.690 |
| 5.0 | 1.748 | 2.087 | 2.534 | 3.145 | 3.539 | 4.016 | 5.347 | 7.619 |
| 10.0 | 1.552 | 1.813 | 2.139 | 2.554 | 2.806 | 3.094 | 3.814 | 4.805 |
| 15.0 | 1.403 | 1.612 | 1.865 | 2.175 | 2.356 | 2.557 | 3.035 | 3.643 |
| 20.0 | 1.284 | 1.458 | 1.663 | 1.906 | 2.045 | 2.197 | 2.547 | 2.972 |
| 25.0 | 1.187 | 1.334 | 1.505 | 1.703 | 1.815 | 1.935 | 2.206 | 2.525 |
| 30.0 | 1.106 | 1.233 | 1.378 | 1.544 | 1.636 | 1.734 | 1.953 | 2.204 |
| 35.0 | 1.036 | 1.148 | 1.274 | 1.415 | 1.493 | 1.576 | 1.757 | 1.961 |
| 40.0 | 0.976 | 1.075 | 1.186 | 1.309 | 1.376 | 1.446 | 1.600 | 1.771 |
| 45.0 | 0.924 | 1.012 | 1.110 | 1.219 | 1.277 | 1.339 | 1.472 | 1.618 |
| 50.0 | 0.877 | 0.957 | 1.045 | 1.142 | 1.194 | 1.248 | 1.365 | 1.492 |
| 55.0 | 0.835 | 0.908 | 0.988 | 1.076 | 1.122 | 1.171 | 1.275 | 1.387 |
| 60.0 | 0.797 | 0.864 | 0.938 | 1.017 | 1.060 | 1.104 | 1.197 | 1.298 |
| 65.0 | 0.763 | 0.825 | 0.893 | 0.966 | 1.005 | 1.045 | 1.130 | 1.221 |

Table 10. Calculated Isothermal Compressibility $\kappa_{T}$ for R410A as a Function of Temperature $T$ and Pressure $p$

| $p / \mathrm{MPa}$ | $\kappa_{\mathrm{T}} / \mathrm{GPa}^{-1}$ |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $T=260 \mathrm{~K}$ | $T=270 \mathrm{~K}$ | $T=280 \mathrm{~K}$ | $T=290 \mathrm{~K}$ | $T=295 \mathrm{~K}$ | $T=300 \mathrm{~K}$ | $T=310 \mathrm{~K}$ | $T=320 \mathrm{~K}$ |
| 1.0 | 3.299 | 4.144 | 5.334 |  |  |  |  |  |
| 3.0 | 3.101 | 3.800 | 4.787 | 6.200 | 7.187 | 8.493 | 12.918 | 23.195 |
| 5.0 | 2.895 | 3.522 | 4.349 | 5.482 | 6.233 | 7.175 | 10.006 | 15.314 |
| 10.0 | 2.511 | 2.986 | 3.569 | 4.303 | 4.750 | 5.271 | 6.629 | 8.647 |
| 15.0 | 2.227 | 2.602 | 3.046 | 3.576 | 3.882 | 4.225 | 5.058 | 6.170 |
| 20.0 | 2.006 | 2.312 | 2.667 | 3.075 | 3.303 | 3.551 | 4.126 | 4.845 |
| 25.0 | 1.829 | 2.084 | 2.377 | 2.706 | 2.885 | 3.076 | 3.504 | 4.012 |
| 30.0 | 1.683 | 1.900 | 2.148 | 2.421 | 2.568 | 2.721 | 3.055 | 3.438 |
| 35.0 | 1.561 | 1.747 | 1.960 | 2.194 | 2.317 | 2.444 | 2.716 | 3.017 |
| 40.0 | 1.458 | 1.619 | 1.804 | 2.007 | 2.113 | 2.222 | 2.450 | 2.694 |
| 45.0 | 1.369 | 1.509 | 1.672 | 1.851 | 1.944 | 2.039 | 2.234 | 2.439 |
| 50.0 | 1.292 | 1.414 | 1.559 | 1.718 | 1.801 | 1.885 | 2.056 | 2.233 |
| 55.0 | 1.225 | 1.330 | 1.460 | 1.604 | 1.679 | 1.754 | 1.907 | 2.061 |
| 60.0 | 1.165 | 1.257 | 1.374 | 1.504 | 1.572 | 1.641 | 1.779 | 1.916 |
| 65.0 | 1.112 | 1.192 | 1.297 | 1.417 | 1.479 | 1.542 | 1.668 | 1.792 |

Table 11. Experimental Critical Data, $p_{c}$ and $T_{c}$, of the System $\mathbf{C O}_{2}$ + R410A

| $x_{\mathrm{CO}_{2}}$ | $p_{\mathrm{c}} / \mathrm{MPa}$ | $T_{\mathrm{c}} / \mathrm{K}$ | $x_{\mathrm{CO}_{2}}$ | $p_{\mathrm{c}} / \mathrm{MPa}$ | $T_{\mathrm{c}} / \mathrm{K}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 0.979 | 7.39 | 306.45 | 0.501 | 6.35 | 328.95 |
| 0.951 | 7.35 | 307.75 | 0.254 | 5.84 | 336.10 |
| 0.919 | 7.32 | 309.03 | 0.100 | 5.30 | 341.05 |
| 0.902 | 7.27 | 311.32 | 0.053 | 5.01 | 343.45 |
| 0.855 | 7.16 | 314.50 | 0.000 | 4.90 | 344.90 |
| 0.746 | 6.97 | 319.86 |  |  |  |

in the ensemble of the isothermal curves is taken as the critical temperature and pressure.

The experimental critical point of the azeotropic mixture R410A was determined by different authors. ${ }^{28,29}$ Nagel and Bier reported the values $T_{\mathrm{c}}=344.75 \mathrm{~K}$ and $p_{\mathrm{c}}=4.93 \mathrm{MPa}$. Kishizawa et al. ${ }^{6}$ determined a slightly lower critical temperature ( 344.48 K ) for R410A by observation of the critical opalescence. Kato et al. ${ }^{29}$ reported the following critical parameters: $T_{\mathrm{c}}=$ 344.63 K and $p_{\mathrm{c}}=4.90 \mathrm{MPa}$. These data deviate less than 0.1 $\%$ in temperature and $0.2 \%$ in pressure from the gas-liquid critical values obtained by our acoustic method: 344.90 K for critical temperature and 4.90 MPa for critical pressure. It can be concluded that the acoustic results measured in our apparatus are in good agreement with the literature values within the combined experimental errors.
$\mathrm{CO}_{2}+$ R410A. The critical line of $\mathrm{CO}_{2}+\mathrm{R} 410 \mathrm{~A}$ mixture was determined over the whole composition range. Acoustic measurements were performed on seven mixtures. The experimental critical parameters, $T_{\mathrm{c}}$ and $p_{\mathrm{c}}$, are shown in Table 11 for each mixture and are plotted in Figure 9. The system exhibits type I fluid phase behavior, according to the classification of van Konynenburg and Scott, ${ }^{30}$ which means that a continuous critical line connects the critical points of R410A and $\mathrm{CO}_{2}$. This
system behaves almost ideally. The projections of critical line are continuous lines connecting the critical points. No maxima or minima in the projections of the critical line are visible. This suggests that R410A should be a useful modifier for $\mathrm{CO}_{2}$. Its relatively low critical temperature ( 344.90 K ) and its miscibility with $\mathrm{CO}_{2}$ make it potentially attractive as modifier for improving solubility in supercritical $\mathrm{CO}_{2}$. As can be seen from Figure 9, the critical pressure of the system goes from $p_{\mathrm{c}}$ of pure $\mathrm{CO}_{2}$, down to 4.90 MPa , the critical pressure of R410A. This means that as R 410 A is added to $\mathrm{CO}_{2}$ the solvation properties of the mixture will have substantially changed, but the critical temperature and critical pressure remain reasonably low. Similar critical behavior was observed in our former study on the $\mathrm{CO}_{2}$ + R404A system. ${ }^{11}$ As it was concluded for R404A, R410A could also conveniently be used to modify $\mathrm{CO}_{2}$.

## Correlation of Data

Experimental critical data were correlated with the PR EOS ${ }^{31}$ that is a cubic EOS based on a one-fluid model:

$$
\begin{equation*}
p=\frac{R T}{\left(V_{\mathrm{m}}-b\right)}-\frac{a(T)}{V_{\mathrm{m}}\left(V_{\mathrm{m}}+b\right)+b\left(V_{\mathrm{m}}-b\right)} \tag{16}
\end{equation*}
$$

In its extension to binary mixtures, the characteristic parameters $a$ and $b$ are calculated using the following van der Waals onefluid mixing rules, which assume random distribution of molecules:

$$
\begin{align*}
& a=x_{1}^{2} a_{1}+2 x_{1} x_{2} a_{12}+x_{2}^{2} a_{2}  \tag{17}\\
& b=x_{1}^{2} b_{1}+2 x_{1} x_{2} b_{12}+x_{2}^{2} b_{2} \tag{18}
\end{align*}
$$



Figure 9. Projections of the critical curve for the $x \mathrm{CO}_{2}+(1-x)$ R410A system: (a) $p-x$ projection. (b) $T-x$ projection.

In these equations $x_{i}$ is the mole fraction of component $i$ ( $i=1,2$ ), and $a_{i}$ and $b_{i}$ are pure component parameters defined by Peng and Robinson. ${ }^{31}$ In particular, $a_{i}(T)=a_{\mathrm{c}, i}\left(T_{\mathrm{c}, i}\right)$ $\alpha_{i}\left(T, \beta_{i}\left(\omega_{i}\right)\right)$, where $\omega_{i}$ is Pitzer's acentric factor of molecule $i$. The combining rules for the binary cross-interaction parameters $a_{12}$ and $b_{12}$ are given by

$$
\begin{align*}
& a_{12}=\left(a_{1} a_{2}\right)^{1 / 2}\left(1-k_{12}\right)  \tag{19}\\
& b_{12}=\frac{1}{2}\left(b_{1}+b_{2}\right)\left(1-l_{12}\right) \tag{20}
\end{align*}
$$

Here $k_{12}$ and $l_{12}$ and parameters describing deviations from, respectively, the geometric and arithmetic mean combining rules. Values of critical temperature, critical pressure and acentric factor ${ }^{32,33}$ for each pure fluid, which are presented in Table 12, constitute, along with the deviation parameters, the model input variables for PR EOS. R410A acentric factor was calculated using the definition of Pitzer ${ }^{34-37}$ and the experimental data $T_{\mathrm{c}}$, $p_{\mathrm{c}}$, and vapor pressures reported by Solvay Technical Data Sheets. ${ }^{38}$ The deviation parameters $k_{12}$ and $l_{12}$ were varied so that the theoretical critical line would give the best fit of the experimental data for each binary system. Although the deviation parameters are often fitted to the experimental critical data using only the equimolar mixture, ${ }^{20}$ we used all the experimental data available for evaluating these parameters. The deviation parameters of $\mathrm{CO}_{2}+\mathrm{R} 410 \mathrm{~A}$ that gave the best fit were $k_{12}=$ 0.0002 and $l_{12}=0.015$. Figure 10 shows the ( $p, T$ ) projection of the critical line for this system using the optimized interaction parameters and the pure component vapor pressures for the


Figure 10. $p-T$ projection of the critical curve for the $x \mathrm{CO}_{2}+(1-x)$ R410A system: - , acoustic data; - , predicted critical curve with PR-EOS; -- , vapor pressure curves for $\mathrm{CO}_{2} ;{ }^{33}-\cdots$, vapor pressure curve for R410A. ${ }^{34}$

Table 12. Parameters of $\mathrm{CO}_{2}$ and R 410 A Used in the Peng-Robinson Equation of State

| component | $p_{\mathrm{C}} / \mathrm{MPa}$ | $T_{\mathrm{C}} / \mathrm{K}$ | $\omega_{i}$ |
| :---: | :--- | :--- | :--- |
| $\mathrm{CO}_{2}$ | $7.38^{35,36}$ | $304.13^{36}$ | $0.239^{37}$ |
| R 410 A | $4.89^{38}$ | $344.95^{38}$ | 0.287 |

binary system. It can be seen from this figure that the PR EOS gives a reasonable agreement with the experimental data for this binary system.

## Conclusions

The thermophysical properties and critical parameters for the alternative refrigerant R410A (50 wt \% of pentafluoroethane $(\mathrm{R} 125)+50 \mathrm{wt} \%$ of difluoromethane (R32)) were investigated using two different acoustic techniques. The critical behavior of the system $x \mathrm{CO}_{2}+(1-x) \mathrm{R} 410 \mathrm{~A}$ was also investigated over the whole composition range. With mole fractions of up to 10 \% of R410A, critical temperature only increases 7 K , and the critical pressure decreases only 0.1 MPa . The critical line shows type I fluid phase behavior. The results were well-correlated with the PR EOS using conventional mixing and combining rules.

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