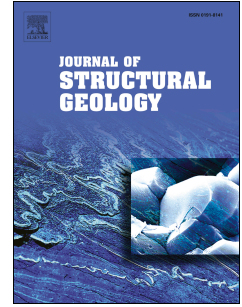


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1 **Analysis of fault bend folding kinematic models and comparison with an analog experiment.**

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24 Abstract:

25 Analog modeling of a flat-ramp-flat fault system was performed and its geometry and
26 displacement field were compared to those of different kinematic models such as classical fault
27 bend folding, fault parallel flow, incline-shear, curvilinear hinge, and backlimb trishear. To obtain
28 the displacement vectors of the analog experiment, a Particle Image Velocimetry was performed.
29 All analyzed kinematic models could explain the general configuration of the fault bend folding.
30 However, only backlimb trishear could represent the geometry, directions of particle displacements,
31 and relations between the displacements' vectors. We propose in this paper that the combination of
32 different asymmetry angles and different apical angles of the backlimb trishear model for each bend
33 in a fault bend fold could be a very versatile and general kinematic model for simulating fault bend
34 folds. Backlimb trishear apical angle can be used to control the shape of the hinges of a fold, while
35 the asymmetry can be used to convolve the velocity of the particles above the fault. Both apical
36 angle and asymmetries different from zero imply thickness changes. Fault bend folds with high
37 inclination forelimbs can be reproduced with high positive asymmetries in the anticline bends of the
38 fault.

39

40 1. Introduction

41 Fault bend folding forms as the result of the movement of a fault block along a non-planar
42 fault surface, which causes the bending of the block and therefore the formation of the fold. This
43 deformation generally occurs in the hanging wall of the fault (Suppe, 1983; Poblet, 2004; Brandes
44 and Tanner, 2014). It is one of the main causes of the folding of the rocks in nature and the object of
45 different types of analog, numerical-mechanical, geometric, and kinematic modeling. Although,
46 mechanical models (either analog or numerical) allow a better understanding of the dynamics of
47 deformation, geometric and kinematic models have a practical utility when constructing complex
48 balanced cross-sections, and that is why they are the target focus of this work. Different geometric
49 and kinematic models (Figure 1) were proposed to simulate fault bend folding (Suppe, 1983; White

50 et al., 1986; Egan et al., 1997; Kane et al., 1997; Cristallini and Allmendinger, 2002), and some of
51 them were implemented in balance cross-section construction software (Contreras, 2002; Cristallini
52 et al., 2021). We use an analog model as a basis to analyze the different kinematic models and to
53 propose variations in the backlimb trishear method (Cristallini and Allmendinger, 2002) that may be
54 useful to simulate flat-ramp-flat fault systems.

55 “Insert Figure 1 here”

56 To produce a fault bend fold in our physical model, appropriate analog materials were used
57 to simulate the upper crust. We employed a flexible sugar paste, which allows us to generate the
58 folding. The analysis of a succession of images, using particle-image velocimetry (PIV), provides a
59 digital visual record of the velocity/displacement vectors during the evolution of the structure. In
60 this work, the shape obtained and the displacement field measured in the analog model of fault bend
61 fold are compared with several geometric and kinematic models. Here we show that backlimb
62 trishear is the model that most faithfully reproduces the geometry, directions of particle
63 displacements, and relations between the displacements’ vectors in different parts of the fold. We
64 also prove that this method is well suited to replicate structures with high dipping forelimbs and that
65 by modifying the asymmetries of the backlimb trishear model, changes in the dipping of the layers
66 involved during folding can be achieved.

67 In the following section, we will first review existing geometric and kinematic models.
68 Next, we will present the analog model performed, and finally, we discuss our overall results.

69 **2. Fault bend folding models**

70 The first quantitative model (here considered as classical fault bend folding model: CFBF)
71 describing the geometry and kinematics of fault bend folds was proposed by Suppe (1983) based on
72 conservation of area and thickness of beds during deformation (Figure 1a). Suppe (1983)
73 formulated the equations that determine both the geometry and kinematics for a fold developed over
74 a fault with a single step or break, as well as for more complex folds formed on ramps with different
75 angles, sheared folds, and hybrid structures (Poblet, 2004; Brandes and Tanner, 2014). The initial

76 assumptions applied for the simplest case are that area is conserved and the thickness of the layers
77 is preserved throughout the evolution of the fold (Suppe, 1983). Therefore, the model ensures that
78 bed length in the slip direction remains constant during deformation. The layers are deformed by
79 flexural slip and axial surfaces are always bisectors of bed bendings (Figure 1a). According to this
80 model, the characteristic shape of a fault bend fold consists of a frontal limb with a greater dip than
81 the backlimb, which remains parallel to the fault's ramp. The evolution of simple step fault bend
82 folds corresponds to two perfectly differentiated stages (Suppe, 1983; Poblet, 2004) known as the
83 lifting of the crest and widening of the crest. It is important to notice that, during the first stage, the
84 slip applied to the hanging wall is not all transmitted forward. In Figure 1a, applied slip is indicated
85 as S_0 and transmitted slip as S_1 , so in this model $S_0 > S_1$. Suppe (1983) CFBF conserve area (in
86 cross-sections), thickness, and line length during deformation.

87 The kinematic field that is associated with the model of kink band migration (Suppe, 1983)
88 was presented by Johnson and Berger (1989). The model proposes that within a simple step
89 structure, 3 velocity domains can be defined based on the fault's geometry. Discontinuities separate
90 these domains and are equivalent to the active axial surfaces previously characterized by Suppe
91 (1983). Vectors are parallel to the lower fault plane in the first domain, then parallel to the ramp in
92 the second domain, and in the third domain, they are parallel to the top fault plane. Hardy (1995)
93 contributes to the development of the kinematic analysis of fault bend folds, describing the
94 horizontal and vertical components of the velocity vectors relying on trigonometric relationships
95 that consider the ramp's dip. Just as in the kink band model the most important operating
96 mechanism is flexural slip (Suppe, 1983; Medwedeff and Suppe, 1997), other models rely on
97 different mechanisms for folding.

98 One of the most commonly used is the fault parallel flow (FPF in this work) proposed by
99 Egan et al. (1997) and Kane et al. (1997). This model (Figure 1b) is based on migration parallel to
100 the fault of the materials of the hanging wall, using axial surface bisectors of fault bends as limits
101 between different velocities. This method states that all particles within the hanging wall move

102 parallel to the fault surface, along virtual flow paths (Ziesch et al., 2014). Subsequent studies on
103 FPF allow calculating the associated deformation in different areas of the structure (Ziesch et al.,
104 2014). Figure 1b shows that slip applied to the hanging wall (S_0) is completely transmitted forward:
105 $S_0 = S_1$. This model conserves only area (in cross-sections) during deformation; thickness and line
106 length are not preserved (see for example thinning of the forelimb in Figure 1b).

107 Another mechanism that can operate in kinematic reconstructions of fault bend folding is the
108 simple shear (Gibbs 1983; 1984) and its derivations (White et al., 1986; White, 1992; Yamada and
109 McClay, 2003). Initially, the method assumed that the hanging wall is deformed by simple shear in
110 vertical planes (Gibbs 1983; 1984). As the angle of shear is vertical, the model is referred to as
111 vertical-shear (Figure 1c). Modifications were subsequently developed and the assumption about
112 the inclination of the shear planes to the vertical is removed (White et al., 1986); this is why the
113 model is commonly known as incline-shear (ISh in this work), where vertical-shear is a special
114 case. The direction of simple shear within the hanging wall block is constant and has a very strong
115 influence on the shape of the resulting fold (White et al., 1986; White, 1992). The slip applied to the
116 hanging wall could be amplified, conserved, or reduced depending on the shear angle. If the shear
117 angle is vertical, the slip is conserved along a complete simple step structure (Figure 1c). This
118 model conserves only area (in cross-sections) during deformation; thickness and line length are not
119 preserved (see for example thinning of backlimb and forelimb in Figure 1c).

120 The theory initially formulated by Suppe (1983) covers exclusively folds formed from faults
121 composed of straight segments with angular breaks, so the modeled examples fail to mimic the
122 traits observed in natural cases. They fail to reconstruct the curved geometry seen in many of the
123 main faults from which the anticlines are generated (Medwedeff and Suppe, 1997). To achieve this
124 feature, Medwedeff and Suppe (1997) propose a model where the main fault has several segments.
125 The length of each segment is reduced, so the fault can be recreated with curved geometry. In this
126 way, the resulting hinge is no longer sharp; on the contrary, it is rounded, which is more consistent
127 with observations from field examples. In turn, each bend generates new axial surfaces that interfere

128 with each other, bringing greater complexity to the structure (Medwedeff and Suppe, 1997). This
129 same idea can also be applied, and simpler, to fault parallel flow and incline-shear models, and is
130 usually used in software cross-section construction. Another modification of the Suppe (1983)
131 CFBF includes the application of basal shear to be able to explain fault bend folds in which
132 backlimb inclination is less than fault dipping (Suppe et al., 2004).

133 Cristallini and Allmendinger (2002) have pointed out that in several analog and mechanical
134 models of fault bend folding formed above faults composed of straight segments with angular
135 breaks, the resulting fold has rounded hinges. These results cannot be explained by classical fault
136 bend folding, neither by fault parallel flow nor incline-shear. To explain these cases, they propose
137 the backlimb trishear model (BLT in this work; Figure 1e, f, and g; Figure 2) where the fold hinge
138 describes soft curvatures in the upper strata while those layers near the fault zone show strong
139 angular breaks (Cristallini and Allmendinger, 2002). This model conserves only area (in cross-
140 sections) during deformation; thickness and line length are not preserved.

141 To explain a similar situation, Tavani et al. (2005) replace axial surfaces represented with
142 straight lines by circular zones that generate the curved geometry seen in fold's layers (Figure 1d).
143 This curvilinear hinge model (CH in this paper) conserves area (in cross-sections), thickness, and
144 line length during deformation, and transmits some shear forward. Tavani et al. (2005) model
145 explains rounded anticline hinges, however, cannot explain rounded syncline hinges.

146 "Insert Figure 2 here"

147 The backlimb trishear model (Cristallini and Allmendinger, 2002) is conceptually analogous
148 to forelimb trishear (Erslev, 1991, Allmendinger, 1998), and presupposes incompressible flow in
149 triangular zones focused on the fault bends. It uses equations of area conservation, similar to those
150 derived for forelimb trishear by Zehnder and Allmendinger (2000), but in these cases applied to the
151 material above a fault bend. Cristallini and Allmendinger (2002) focused their paper on the
152 backlimb of a fold and named their model as "backlimb trishear" (Figure 2). However, the idea (and
153 of course the equations) can be applied to any bend in a fault and can be used to explain syncline

154 and anticline hinges (Figure 1 e, f, and g). The backlimb trishear, in addition to hanging wall slip
155 and fault bend angle, has basically two variables, the backlimb trishear apical angle, and the
156 asymmetry. The second is measured with respect to the fault bend bisector and is positive forward
157 and negative backward. Cristallini and Allmendinger (2002) showed that a symmetrical backlimb
158 trishear zone does not produce the variation of the applied slip versus the transmitted slip. However,
159 asymmetrical zones produce slip variations. Changing the backlimb trishear asymmetry, the model
160 can satisfy the slip variations of classical fault bend folding (Figure 1e), fault parallel flow (Figure
161 1f), or incline-shear (Figure 1g).

162 **3. Analog model methodology**

163 Intending to analyze and test different kinematic models of fault bend folding, we perform
164 an analog model to obtain the displacement field during the formation of simple step fault bend
165 folds. In this way, it is possible to evaluate and compare the displacement field and the resulting
166 geometries with those of the investigated kinematic models. The experiment focuses on evaluating
167 the vectors for the first stage of fold growth, where lifting of the crest occurs.

168 The analog model technique is practical and simple for obtaining displacement fields during
169 deformation. Vectors of displacement are obtained by particle image velocimetry (Sveen, 2004), a
170 methodology widely used in geological process studies (Kincaid and Griffiths, 2003; Boutelier and
171 Cruden, 2013; Strak and Schellart, 2014; Schellart and Strak, 2016) performed with PIVlab-
172 MATLAB program (Thielicke and Stamhuis, 2014). The results obtained were compared with the
173 previously analyzed kinematic models (Figure 1). The technique of analog modeling is useful since
174 it allows the incorporation of a mechanical framework into the kinematic method.

175 *3.1. Analog model setup*

176 To simulate the stage of the lifting of the crest in a simple step fault bend fold, we used an
177 experimental setup consisting of a footwall represented by a rigid, non-deformable wedge and a
178 deformable hanging wall represented by a layered plastic material (Figure 3). To meet the
179 objectives of this experiment, a material that would not fracture or fail was needed. Cohesionless

180 materials as sand, traditionally used in experiments of deformation of the upper crust (Cristallini et
181 al., 2009; Ritter et al., 2016; Marshak et al., 2019), do not meet these characteristics. The models
182 required a cohesive material being able to simulate the strata that constitute a sedimentary cover
183 involved in the folding, where no fractures are desired. For this reason, we used sheets of sugar
184 paste to model the hanging wall. The preparation has a density of 1.27 g/cm^3 and a viscosity equal
185 to $2.2 \times 10^7 \text{ Pa s}$, being this value similar to plasticines widely used as analog materials for
186 experimental setups (Schöpfer and Zulauf, 2002; Zulauf and Zulauf, 2004). The sheets are separated
187 from the bottom of the box, the metal wedge, and between them by low friction surfaces. To avoid
188 the formation of voids and to approach the challenging scaling conditions, all the experiment is run
189 inside a biaxial loading cell like that proposed by Bazalgette and Petit (2007). To fill the spaces
190 above the layers of sugar paste, dough made with salt, flour, and water (density $\square 1.29 \text{ g/cm}^3$ &
191 viscosity $\square 1.2 \times 10^5 \text{ Pa s}$) was used. This mass was placed at the top reaching 9 cm in height,
192 separated by a plastic film that acts as a moisture barrier. The dough is used to compress the entire
193 model, increasing vertical pressure over the sugar sheets (Bazalgette and Petit, 2007), and causing
194 the layers of sugar paste to accommodate by folding to the movement of the rigid wedge. As the
195 box is closed, this material allows an increase in the confining stress and thus inhibits the separation
196 of the sugar paste from the fault block.

197 “Insert Figure 3 here”

198 To ensure that the deformation occur on top of the wedge, it was decided not to move the
199 hanging wall, as in a classical fault bend fold, but move the footwall wedge instead. The sugar paste
200 layers were cut with different lengths to be in close contact with the ramp of the metal wedge to
201 simulate the fault. However, they cannot be extended to the right side of the experiment because a
202 classical fault bend fold transmits part of the applied slip forward by the upper plane. To simulate
203 this situation, the layers of sugar paste are constructed shorter than the box, and the space that is not
204 occupied by them is filled by a colorless silicone with non-Newtonian behavior with a density of
205 0.97 g/cm^3 and a viscosity of $5 \times 10^4 \text{ Pa s}$ at 20° C (Weijermars, 1986). This idea is similar to that

206 used by Chester et al. (1991) in the apparatus to simulate a fault bend fold and is needed to
207 represent this type of structure.

208 Plane strain condition of the experiment was ensured with continuous observation. The
209 model was photographed on both sides of the deformation box, mounted with two opposite acrylic
210 walls for this purpose. The apparatus also has a motor-driven piston to compress the materials
211 arranged inside. The experiment carried out was 47 cm long by 15 cm wide and 19 cm high (Figure
212 3). Inside it and in contact with the piston, a rigid wedge of 30° was placed. The wedge is 22 cm
213 long at its base and 14 cm on the upper fault flat and its ramp is 9 cm long. This device simulates
214 the motion of the hanging wall past the fault bends (Zanon and Gomes, 2019). At the base of the
215 experiment, only 25 cm of sugar paste was placed while at the top were placed 37 cm and 10 cm of
216 transparent silicone in contact with the piston (Figure 3). Above the sugar paste and the silicon, the
217 dough was used to fill the box and increase vertical load over the experiment.

218 The model was compressed for 67 minutes, at 10 cm/hour, reaching a total of 11 cm of
219 shortening and forming a fault bend fold. As the structure is generated, photographs were taken
220 perpendicular to the direction of motion of the piston on both sides of the model to follow the
221 evolution of the fold. Previous trials concluded that the deformation observed through the sidewalls
222 of the box is representative of the internal deformation within the models and plane strain can be
223 assumed. A total of 67 images were obtained, one per minute. These photographs were
224 subsequently processed using the Irfanview TM (Skiljan, 2012) software to crop the area of interest
225 and generate the serial images. These images were analyzed with the PIVlab-MATLAB program
226 (Thielicke and Stamhuis, 2014) to obtain the kinematic vectors that illustrate the displacement field
227 of particles that generate the fold and the evolution during the lifting of the crest.

228 This experiment is a qualitative analog for a two bend fault bend fold system; it is not an
229 accurately scaled physical model (Hubbert, 1937). It is intended to compare shapes and relative
230 displacement and velocity relations with kinematics models. In this work, we focused on the

231 different domains of the displacement field during the evolution of a fault bend fold rather than the
232 absolute values of the displacement vectors.

233 *3.2. Particle image velocimetry:*

234 There are numerous programs to carry out a particle image velocimetry (Adam et al. 2002,
235 Adam et al., 2005; Schellart and Strak, 2016), but we selected software PIVlab-MATLAB
236 (Thielicke and Stamhuis, 2014) because it is easy to implement and enables complex graphics of
237 both the displacement vectors and their corresponding PIV-derived parameters like the magnitude
238 of displacement velocity (Krýza et al., 2019).

239 To calculate the vectors this tool divides each of the images into user-defined areas, of a
240 certain number of pixels to be analyzed. The image should be calibrated indicating both the actual
241 distance (in mm) between two points in the photo and the time interval (in ms) between 2
242 successive photos. In this experiment, each pixel in the photographs represents 0.02 mm of the
243 analog model and the time interval was equal to 60 s. In each defined area, the program compares
244 the pixels for two successive images, detecting differences that are attributable to the movement and
245 plotting the vectors. The presented displacement vectors are calculated on the basis of redistribution
246 of the pixels between the photographs taken 60 s apart, representing the total displacement over that
247 interval. After processing, validation of the vectors is performed, crossing out outliers considering
248 that maximum velocity is the one of the motor-driven piston.

249 To concentrate the deformation just over the basal ramp, in this experiment the aluminum
250 wedge that represents the footwall of the fault bend fold, worked as a piston, and the hanging wall
251 was passively deformed. Therefore, to compare the results of these experiments with classical fault
252 bend folds, the uniform displacement of the aluminum wedge was subtracted from the obtained
253 vector field. The resultant field was analyzed and plotted in figures 6, 7, and 8.

254 **4. Geometric and kinematic analog model results**

255 *4.1. Comparing fold shape*

256 We use the final stage of the analog model to compare the resulting fold shape with those of
257 different kinematic models (Figure 4). The best fit (blue line in Figure 4) was visually made on the
258 yellow highlighted layer of the experiment. Figure 4a compares the analog model with classical
259 fault bend folding (CFBF; Suppe, 1983), Figure 4b with the fault parallel flow model (FPF; Egan et
260 al., 1997; Kane et al., 1997), Figure 4c with a special case of vertically incline-shear, where the
261 inclination of the shear planes is vertical (ISh; White et al., 1986), Figure 4d with curvilinear hinge
262 model (CH; Tavani et al., 2005) and Figure 4e with a special case of backlimb trishear (BLT;
263 Cristallini and Allmendinger, 2002). In a quick view, all the models explain the general geometry of
264 the fold of the analog experiment. However, there are differences, and some models explain better
265 some features than others. For example, CFBF, FPF, and ISh fail to explain the curvilinear shape of
266 the fold, while CH and BLT represent very well this feature for the anticline, but CH does not copy
267 the curvilinear shape of the synclines. ISh model fits very well the area covered by the fold,
268 however, like CFBC, FPF, and CH, it does not fit the slip over the footwall ramp with that of the
269 analog model.

270 “Insert Figure 4 here”

271 To analyze backlimb trishear fitting, we tested different trishear apical angles and
272 asymmetries (Figure 5). In the initial code of Cristallini and Allmendinger (2002) both parameters
273 had to be the same for all the fault bendings. In Figure 5 we show the results of different
274 asymmetries, using 30° as trishear apical angle. If we compare the resulting curves with the yellow
275 highlighted bed, we can see that -10° asymmetry works better for the backlimb while $+8^\circ$
276 asymmetry works better for the forelimb. For this work, we use the development version of Andino
277 3D software (Cristallini et al., 2020) to modify the original code of Cristallini and Allmendinger
278 (2002) to allow the use of different asymmetries and apical angles for each bend in the fault. In this
279 way, we can produce a better fitting of the yellow highlighted bed using asymmetries of -10° for the
280 backlimb bend and $+8^\circ$ for the forelimb bend (Figure 4e).

281 “Insert Figure 5 here”

282 Natural anticlines with flat-ramp-flat geometry, those in which the fault ramp did not
283 broaden to the top but developed an upper flat in the middle of the stratigraphic column can be
284 comparable with our analog model, in particular, if the sedimentary cover involved in the folding
285 does not develop major secondary faults.

286 *4.2. Comparing displacements and velocities*

287 To trace the movement of the particles in the experiment, we use PIVlab-MATLAB
288 (Thielicke and Stamhuis, 2014). The results can be seen in Figure 6 for three steps with 1.25, 2.5
289 cm, and 4.8 cm of applied slip. The blue vectors in figures 6a, b and c show the incremental
290 displacement field calculated by the PIV. According to the kinematic field, two domains of rotation
291 defining triangular geometries can be recognized, and the movement of particles is concentrated
292 inside them. Figures 6d, e, and f are color maps of the slip vectors direction for the corresponding
293 displacement field; a progressive rotation along the fault bending zone is outlined. The yellow
294 dashed lines of Figure 6 represent the backlimb trishear zones adjusted to the analog model. These
295 triangular zones fit very well the distortion zones seen in the displacement field (Figure 6). The
296 displacement vectors are initially composed of a single horizontal component, V_x . When particles
297 reach the backlimb rotation zone, the vertical component of the displacement vectors increases as
298 the experiment progresses (Figures 6d, e, and f). After that, the displacement vectors remain rather
299 parallel to the surface of the ramp (metal wedge), until particles enter the forelimb rotation zone
300 where they progressively lose the vertical component V_y . Finally, displacement vectors are
301 composed once again by horizontal vectors. The displacement vectors above the ramp are rather
302 parallel to it (Figures 6a and 6b). In advanced stages of the model, the displacement vectors are not
303 completely parallel to the fault, having an angle slightly bigger (Figure 6c). This happens perhaps,
304 because in the final stage of the experiment, the resulting structure, moves a little away from a
305 theoretical fault bend fold, and a smooth lift-off is beginning. Because of this, we considered that
306 the stages represented in figures 6a and b are the most appropriate to make a detailed comparison
307 with a kinematic model.

308 Figure 7 shows a comparison between the displacement vectors calculated by PIV in the
309 experiment (blue arrows) and those calculated by the backlimb kinematic model using -10° and $+8^\circ$
310 asymmetries of backlimb and forelimb respectively and an apical angle of 30° (same model as
311 Figure 6b and 6e). There is a very good agreement between both displacement fields, even in the
312 rotation zones above the fault bendings (Figure 7).

313 “Insert Figure 6 here”

314 “Insert Figure 7 here”

315 To compare the mean velocity vectors, three windows of the experiment section were
316 selected to calculate the average velocity magnitudes outside of the triangular areas where vector
317 rotations take place (Figure 8): one over the basal plane (A), one over the ramp (B), and one over
318 the upper plane (C; Figure 8). The average velocities are 6.8 cm/h, 6.19 cm/h and 5.61 cm/h,
319 respectively. In the same figure, a table shows the predicted velocities for the analyzed kinematic
320 models. The decrease in velocity of B with respect to A can only be explained by the backlimb
321 trishear model (BLT) using the same asymmetries as in figures 4e and 7 (backlimb asymmetry -10°).
322 The decrease in velocity of C with respect to B can be explained by classical fault bend folding
323 (CFBF), curvilinear hinge model (CH), and backlimb trishear (BLT with $+8^\circ$ of forelimb
324 asymmetry). However, the BLT shows the best fit between the velocity magnitudes.

325 “Insert Figure 8 here”

326 To accurately compare the analog model to the different theoretical kinematic models, in
327 Figure 9, we plotted the subtraction of kinematic models velocities to the analog model velocities.
328 Backlimb trishear (BLT) model is the one with the least differences with respect to the analog
329 model, for both the horizontal and vertical components of the velocity vectors (V_x and V_y).

330 “Insert Figure 9 here”

331 To facilitate the application of the equation proposed by Cristallini and Allmendinger (2002)
332 to calculate velocity changes across fault bends, we develop the nomogram in Figure 10, where the
333 resultant velocity after a bend can be calculated with respect to a normalized to 1 input velocity

334 (velocity applied before the bend), the angle between both sections of the fault (φ) and the
335 asymmetry of the backlimb trishear zone (α). The blue curves are for syncline bending of the fault
336 (positives φ), while the orange curves are for anticlinal bendings (negatives φ). The curves for $\varphi =$
337 $+30^\circ$ and $\varphi = -30^\circ$ shown in black are those used for the example of this paper. The blue point
338 corresponds to $\alpha = -10^\circ$ asymmetry of the backlimb adjusted to the experimental fold (Figures 4e
339 and 8) and the red point to the $\alpha = +8^\circ$ asymmetry adjusted to the forelimb (Figures 4e and 8). The
340 resultant velocity $V1$ is calculated as fractions of the input velocity normalized to 1 ($V0 = 1$). This
341 means that an output velocity of $V1 = 1$ implies that there is no change in velocities. Values of $V1 <$
342 1 implies a reduction of velocity and $V1 > 1$ implies an increase. This graph allows sustaining that
343 the BLT model fits the experiment well.

344 "Insert Figure 10 here"

345 5. Discussion

346 The analog simulation described in this work does not represent the generality of the fault
347 bend folds, but it serves to analyze and compare the different kinematic models. We find that all the
348 analyzed kinematic models can broadly explain the fold geometry developed in the experiment
349 (Figure 4). However, backlimb trishear (BLT) is the only one that can mimic accurately the
350 geometry (Figure 4e), directions of particle displacements (Figure 7), and relations between the
351 modulus of the velocity vectors (Figure 8). This is because BLT is the most flexible of the analyzed
352 kinematic models. With the trishear apical angle, the sharpness of the deformation zones above the
353 fault bends can be controlled, while the asymmetries variations can achieve different inclinations of
354 the forelimb and changes in thickness. Moreover, when plotting the slip vectors directions in Figure
355 6, the change in their angle is gradual and occurs along a triangular shaped rotation zone. These
356 results fit well with the backlimb trishear model (BLT).

357 Furthermore, by subtracting the vertical and horizontal components of the velocity vector
358 from the fields proposed in theoretical kinematic models from the field obtained for the analog
359 model, the backlimb trishear model (BLT) is the one that presents the smallest differences.

360 Therefore, it is possible to state that this model is the one that most accurately represents the fold
361 generated in the laboratory and its kinematic evolution. It is postulated that this may be due to the
362 flexibility of the backlimb trishear model (BLT), where a wide range of geometries can be
363 represented from modifications in the aforementioned parameters. The nature of the materials used
364 for the analog model support this conclusion: the sugar paste does not break during deformation, but
365 distributes along the rotation zone presented in BLT model. The same could happen with other
366 cohesive materials such as clay, while coarser granular materials such as dry sand do not develop
367 progressive rotation zones, being probably best represented with other theoretical models.

368 Cristallini and Allmendinger (2002) focused their work on explaining the geometry of the
369 backlimb in a fault bend fold. However, as we pointed before, their equations are more flexible and
370 can be applied to any bend in the fault surface. We modify their original code to enable the use of
371 independent backlimb trishear apical angles and asymmetries for each bend in the fault. In
372 Cristallini and Allmendinger (2002), the authors compare the BLT model with one of the
373 experiments of Chester et al. (1991). However, they only could compare the backlimb of the fold,
374 because of the limitations of the code. Now, we can show a complete comparison of the same fold
375 (Figure 11). There is a very good fit using a backlimb trishear apical angle of 30° and asymmetries
376 of -10° and $+35^\circ$ for backlimb and forelimb respectively (Figure 11).

377 “Insert Figure 11 here”

378 One of the restrictions of geometric and kinematic models of fault bend folding is their
379 ability to represent highly dipping forelimbs. However, this can be solved by adjusting the
380 asymmetry parameter of the BLT model. Figure 12 represents the comparison of one of the models
381 of Chester et al. (1991) with a BLT simulation. In this case, with a forelimb asymmetry of $+50$,
382 forelimb dipping of 80° can be achieved.

383 “Insert Figure 12 here”

384 Finally, based on the analyzes carried out in this work and on cited references, it is clearly
385 observed that most of the described fault bend models (CFBF, FPF, and ISh) imply velocity vectors

386 parallel to the fault, where the only differences are the boundary between domains and the
387 magnitude of the velocity vectors. If we consider the simple step structure as a system of a backlimb
388 fold and a forelimb fold, all these models can be visualized as incline-shear cases, where the field
389 boundaries are in the direction of shear. When the boundary is established symmetrically
390 concerning the fault bend (as a bisector) the velocity magnitudes are conserved. If the boundary is
391 not the bisector of fault bend, the velocity is not preserved on either side of the axial surface. With
392 positive asymmetries, the velocity is incremented after the boundary in a syncline bend and
393 decremented in an anticline bend, while with negative asymmetry the opposite occurs. Fault parallel
394 flow models (FPF) give rise to a symmetrical position of field boundaries with respect to the fault
395 bends and therefore velocities magnitudes are conserved above each fault bend (Figure 1b). In
396 contrast, in the first stage of a classical fault bend folding (CFBF), the axial surfaces are oriented to
397 preserve bedding thickness, and therefore the forward active axial surface does not bisect the fault
398 bend, causing velocity not to be preserved (Figure 1a). For the incline-shear model (ISh), the
399 velocity magnitudes are generally not conserved for each fault bend, and they are incremented or
400 decreased depending on the asymmetry. In the special case of vertical-shear, although the slip is not
401 conserved for each fault bend, it is conserved for the complete system of a simple step structure
402 (Figure 1c). Backlimb trishear (BLT) can be applied to all the previous models (FPF, CFBF, or ISh)
403 just to add progressive rotation to the limbs and to increase the curvilinear geometry of the folds,
404 distributing deformation within a triangular shape shear zone.

405 Although in this article we were able to simulate a fault bend fold using the BLT model,
406 more work still needs to be done to determine which are the mechanical conditions that control
407 asymmetry and apical angle. Although the discrete-elements model of Hardy and Finch (2007) and
408 the analog models of Bazalgette and Petit (2007) were not made to analyze the BLT, they may
409 suggest that the apical angle depends on the mechanical stratigraphy and friction between beds.
410 With a strongly layered mechanical stratigraphy or very low friction between beds, a parallel layer
411 mechanism is favored and consequently low apical angles in the BLT model. Contrary, the high

412 coupling between beds or a weakly layered mechanical stratigraphy could favor high BLT apical
413 angles. However, these relations need to be demonstrated, and others have to be found concerning
414 the asymmetry. For example, in our experience, the folding above most syncline bends of faults can
415 be described with negative asymmetries in the BLT model, but we have no conclusion about the
416 mechanical causes of this.

417 **6. Conclusions**

418 An analog model made in the laboratory is described and processed to derive a particle
419 image velocimetry. The generated displacement vectors illustrate the migration of materials as a
420 fault bend fold evolves. We use this example as a trigger to analyze different geometric and
421 kinematic models: CFBF (classical fault bed folding), FPF (fault parallel flow), ISh (incline-shear),
422 CH (curvilinear hinge model), and BLT (backlimb trishear). All models can explain the bulk
423 displacements of fault bend folding. However, only BLT can represent the geometry (Figure 4e),
424 directions of particle displacements (Figure 7), and relations between the velocity vector fields
425 (Figure 8).

426 We propose that the combination of different asymmetry angles and different apical angles
427 of BLT model for each bend in a fault bend fold could be a very versatile and general kinematic
428 model for describing these types of structures. BLT apical angle can be used to control the shape of
429 the hinges of a fold, while the asymmetry can be used to convolve the velocity of the particles
430 above the fault. Both apical angle and asymmetries different from zero imply thickness changes.
431 BLT mode ensures the conservation of area (in a section) during deformation, even when the
432 asymmetry and apical angle are variable for the different bends of the fault. Fault bend folds with
433 high inclination forelimbs can be reproduced with high positive asymmetries in the anticline bends
434 of the fault.

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558 **Figure Captions:**

559 **Figure 1:** Different geometric-kinematic models for a single step fault bend fold. **a.** Suppe
 560 (1983) fault bend folding model (classical fault bend model in this work: CFBF). **b.** Fault parallel
 561 flow model (FPF) from Egan et al. (1997). **c.** Incline-shear model (ISh) based on White et al.
 562 (1986). In this case, vertical shear indicates that the shear angle of incline-shear model is vertical. **d.**
 563 Curvilinear hinge model (CH) from Tavani et al. (2005). **e., f.** and **g.** Backlimb trishear (BLT) from
 564 Cristallini and Allmendinger (2002) with asymmetries that satisfy those of CFBF, FPF, and ISh
 565 respectively.

566 **Figure 2:** **a.** Backlimb trishear (BLT) implies progressive rotation of the beds over an
 567 angular fault bend. The angle 2ϕ represents the apical angle and V_0 and V_1 the velocities on either
 568 side of the triangular zone. θ is the dip angle for the fault. **b.** Definition of the asymmetry angle (α);
 569 in (a) triangular zone is symmetric ($\alpha=0$). Modified from Cristallini and Allmendinger (2002).

570 **Figure 3:** Sketch of the experiment pointing out the materials used. The wedge has 30° ,
 571 simulating the ramp. The dough consists of salt, flour and water. The black arrows indicate the
 572 direction and velocity of the moving piston.

573 **Figure 4:** Final stage of the hanging wall in the analog model compared to different
 574 kinematic models. The best fit (blue line) was visually made on the yellow highlighted layer of the
 575 experiment **a.** Suppe (1983) fault bend folding model (CFBF). **b.** Fault parallel flow model (FPF)
 576 from Egan et al. (1997). **c.** Incline-shear model (ISh) based on White et al. (1986). **d.** Curvilinear
 577 hinge model (CH) from Tavani et al. (2005). **e.** Backlimb trishear (BLT) from Cristallini and
 578 Allmendinger (2005). The curvilinear shape of the fold can be only obtained using the curvilinear
 579 hinge model or backlimb trishear.

580 **Figure 5:** Curves obtained from applying backlimb trishear, using 30° as apical angle and
 581 different asymmetries (see figure inset color code). Comparing with the fold shape of the analog
 582 model (using the yellow highlighted bed as reference), -10° asymmetry works better for backlimb
 583 while $+8^\circ$ asymmetry works better for forelimb.

584 **Figure 6:** Total component of the displacement field represented with blue vectors. **a.**
 585 Photograph of the experiment with 1.25 cm of applied slip. **b.** Photograph of the experiment with
 586 2.5 cm of applied slip. **c.** Photograph of the experiment with 4.8 cm of applied slip. **d., e., and f.**
 587 Color map graphics of slip vectors direction measured anticlockwise from the x-axis. Yellow
 588 dashed lines represent the backlimb trishear zones adjusted to the analog model.

589 **Figure 7:** Comparison between displacement vectors obtain from the PIV analysis (blue
 590 arrows) performed from the serial images of the analog model and those obtained after applying
 591 backlimb kinematic model (black arrows) using -10° and $+8^\circ$ asymmetries for backlimb and
 592 forelimb, respectively, and an apical angle of 30° . The black dashed lines represent the trishear
 593 zones for the backlimb trishear model. Total component of the displacement field represented with
 594 blue vectors corresponds to 2.5 cm of applied slip.

595 **Figure 8:** **a.** Table presenting different kinematic models and their velocities for the
 596 following regions: A. above basal plane. B. above plane over the ramp and C. above the upper
 597 plane. Kinematic models used are: classical fault bend folding (CFBF), fault parallel flow (FPF),
 598 vertical shear (ISh), curvilinear hinge model (CH) and backlimb trishear (BLT). **b.** Photograph of
 599 the experiment with 2.5 cm of applied slip and the total component of the displacement field
 600 represented with blue vectors. Yellow dashed lines represent the backlimb trishear zones. White
 601 rectangles illustrate the sectors from which the average velocity is calculated.

602 **Figure 9:** Resultant from the subtraction of kinematic models velocities to the analog model
 603 velocities. Kinematic models used are: classical fault bend folding (CFBF), fault parallel flow
 604 (FPF), vertical shear (ISh), and backlimb trishear (BLT) using -10° and $+8^\circ$ asymmetries for
 605 backlimb and forelimb, respectively, and an apical angle of 30° . The left column shows the

606 subtraction of horizontal velocity components (V_x) and the right column, the subtraction of vertical
 607 velocity components (V_y). Note that the BLT model is the one with the least differences with
 608 respect to the analog model.

609 **Figure 10: a.** Scheme showing resultant velocity (V_1) after a fault bend, calculated with
 610 respect to a normalized to 1 input velocity (V_0), the angle between both sections of the fault (φ) and
 611 the asymmetry of the backlimb trishear zone (α). V_0 and V_1 are the velocities on either side of the
 612 triangular zone. φ is the angle between both sections of the fault. α is the angle of asymmetry of the
 613 backlimb trishear zone - the angle between the bisector of the fault bend angle (in a fine black
 614 straight line) and the bisector of the apical angle of backlimb trishear (BLT axial line, dashed). 1.
 615 Syncline bending of the fault (positives φ). 2. Anticlinal bending (negatives φ) (Modified from
 616 Cristallini and Allmendinger 2002). **b.** Curves showing resolution for Cristallini and Allmendinger
 617 (2002) velocity variation equation. Blue curves are for syncline bending of the fault (positives φ),
 618 while the orange curves are for anticlinal bendings (negatives φ). In the analog model, the angle of
 619 the fault was 30° (black curves). Blue point corresponds to $\alpha = -10^\circ$ asymmetry of the backlimb
 620 adjusted to the experiment fold (Figures 4e and 7). Red point corresponds to the $\alpha = +8^\circ$ asymmetry
 621 of the forelimb adjusted to the experiment fold (Figures 4e and 7).

622 **Figure 11:** Comparison between analog model from Chester et al. 1991 and a backlimb
 623 trishear with 30° of apical angle, $\alpha = -10^\circ$ asymmetry for the backlimb and $\alpha = +35^\circ$ asymmetry for
 624 the forelimb.

625 **Figure 12:** Backlimb trishear model reconstruction using ANDINO 3D software. The
 626 geometry obtained in the analog model with a high dipping forelimb can be reproduced. **a.**
 627 Backlimb trishear model with 40° of apical angle, $\alpha = -20^\circ$ asymmetry for the backlimb and $\alpha =$
 628 $+50^\circ$ asymmetry for the forelimb. **b.** Analog model from Chester et al. 1991.

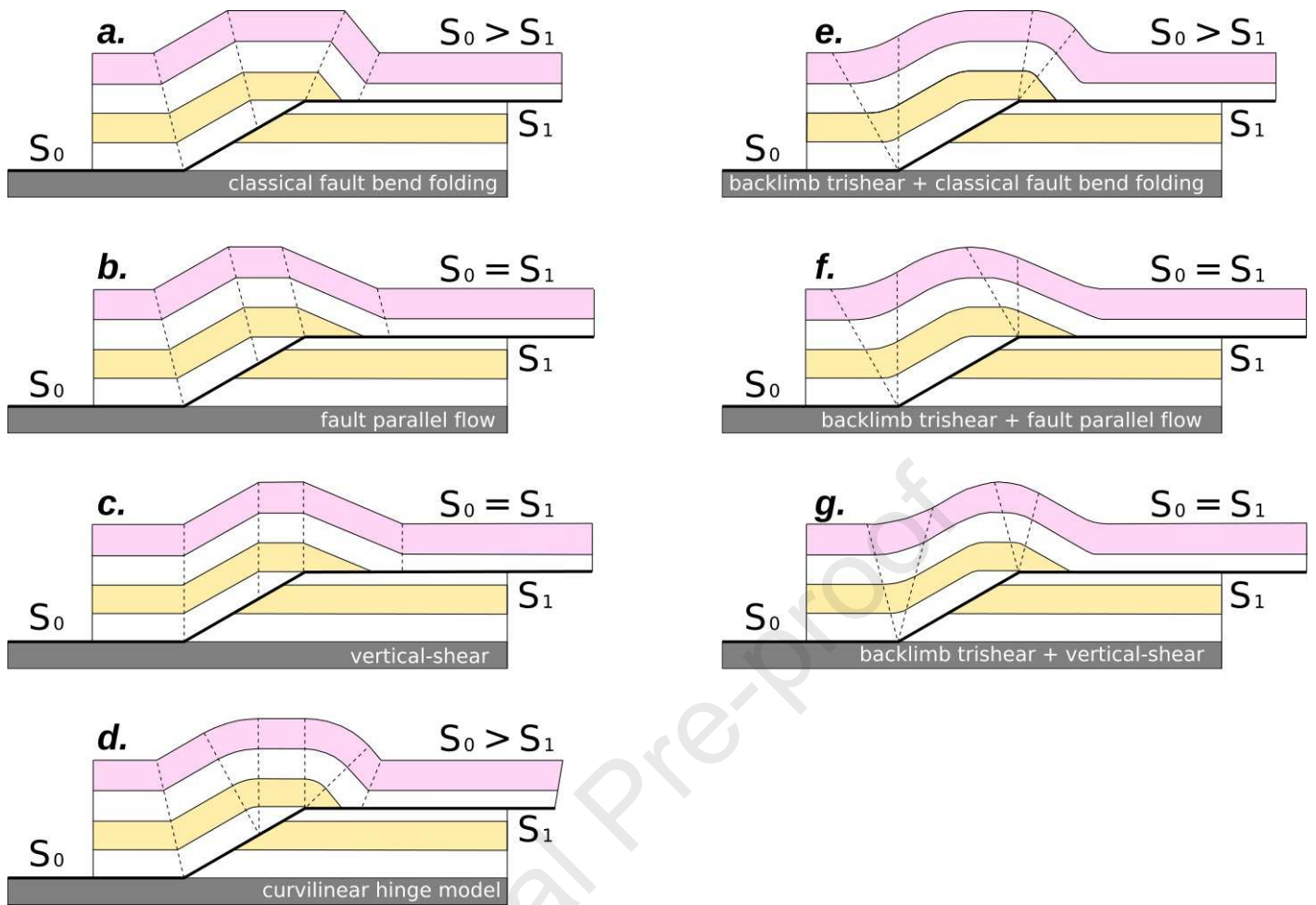


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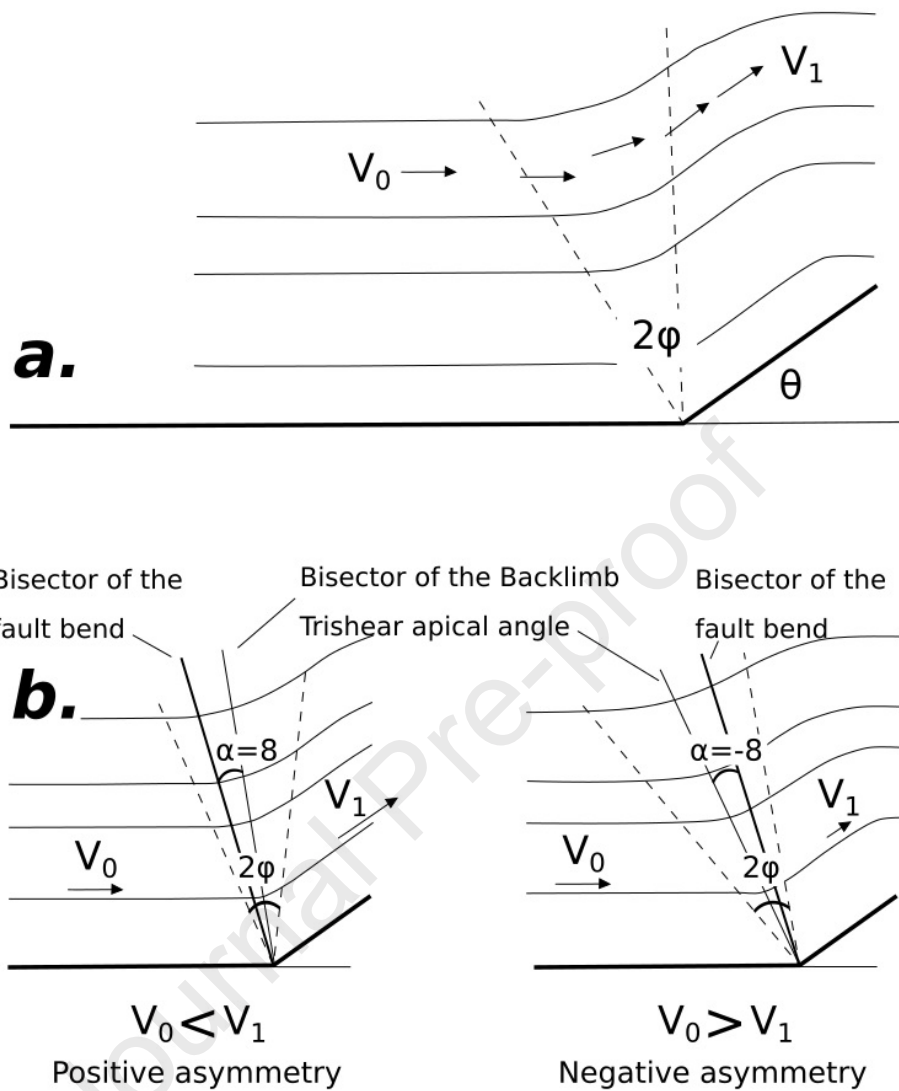


Figure 2: **a.** Backlimb trishear (BLT) implies progressive rotation of the beds over an angular fault bend. The angle 2ϕ represents the apical angle and V_0 and V_1 the velocities on either side of the triangular zone. θ is the dip angle for the fault. **b.** Definition of the asymmetry angle (α); in (a) triangular zone is symmetric ($\alpha=0$). Modified from Cristallini and Allmendinger (2002).

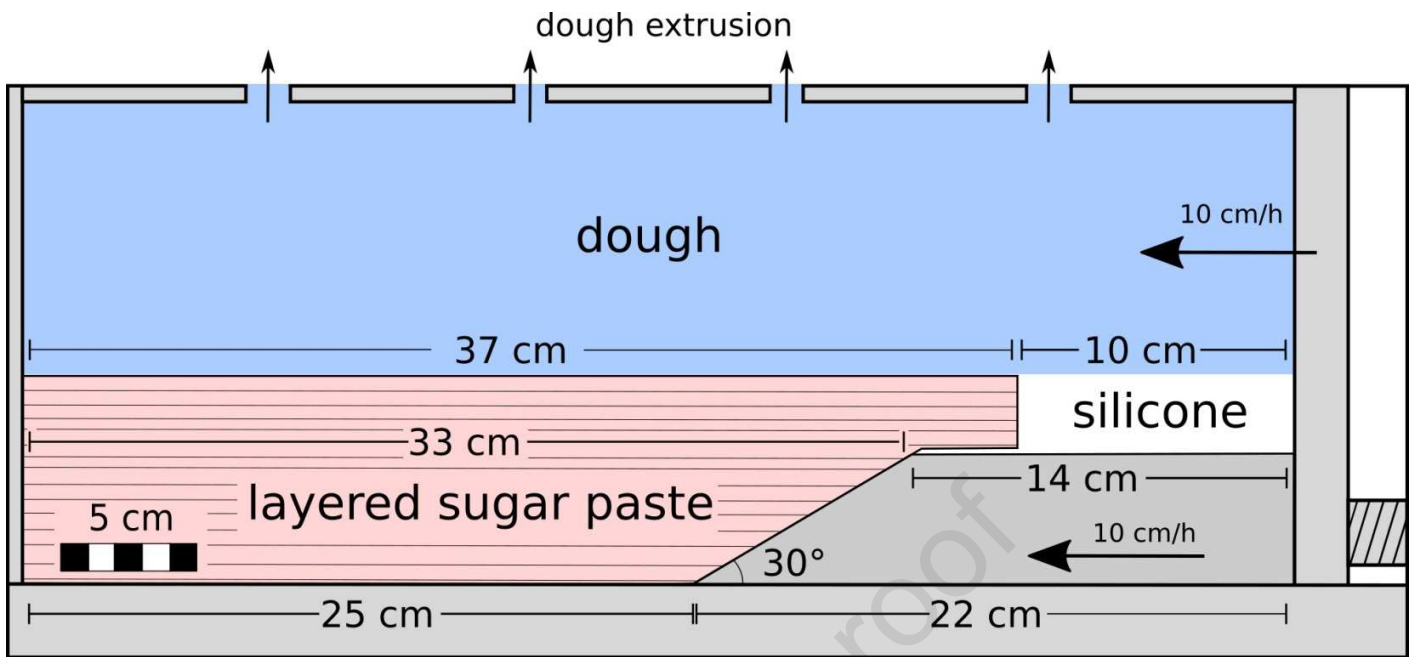


Figure 3: Sketch of the experiment pointing out the materials used. The wedge has 30° , simulating the ramp. The dough consists of salt, flour and water. The black arrows indicate the direction and velocity of the moving piston.

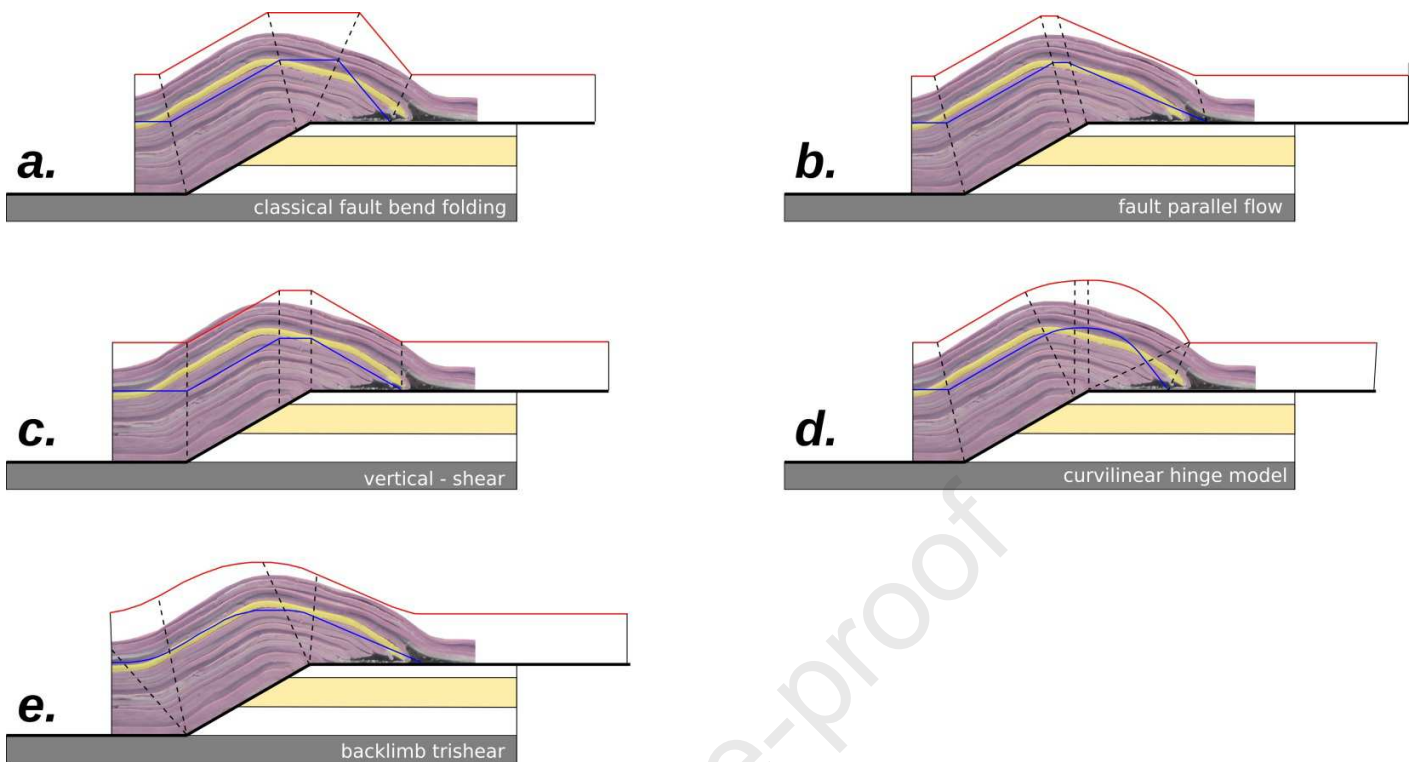


Figure 4: Final stage of the hanging wall in the analog model compared to different kinematic models. The best fit (blue line) was visually made on the yellow highlighted layer of the experiment **a.** Suppe (1983) fault bend folding model (CFBF). **b.** Fault parallel flow model (FPF) from Egan et al. (1997). **c.** Incline-shear model (ISh) based on White et al. (1986). **d.** Curvilinear hinge model (CH) from Tavani et al. (2005). **e.** Backlimb trishear (BLT) from Cristallini and Allmendinger (2005). The curvilinear shape of the fold can be only obtained using the curvilinear hinge model or backlimb trishear.

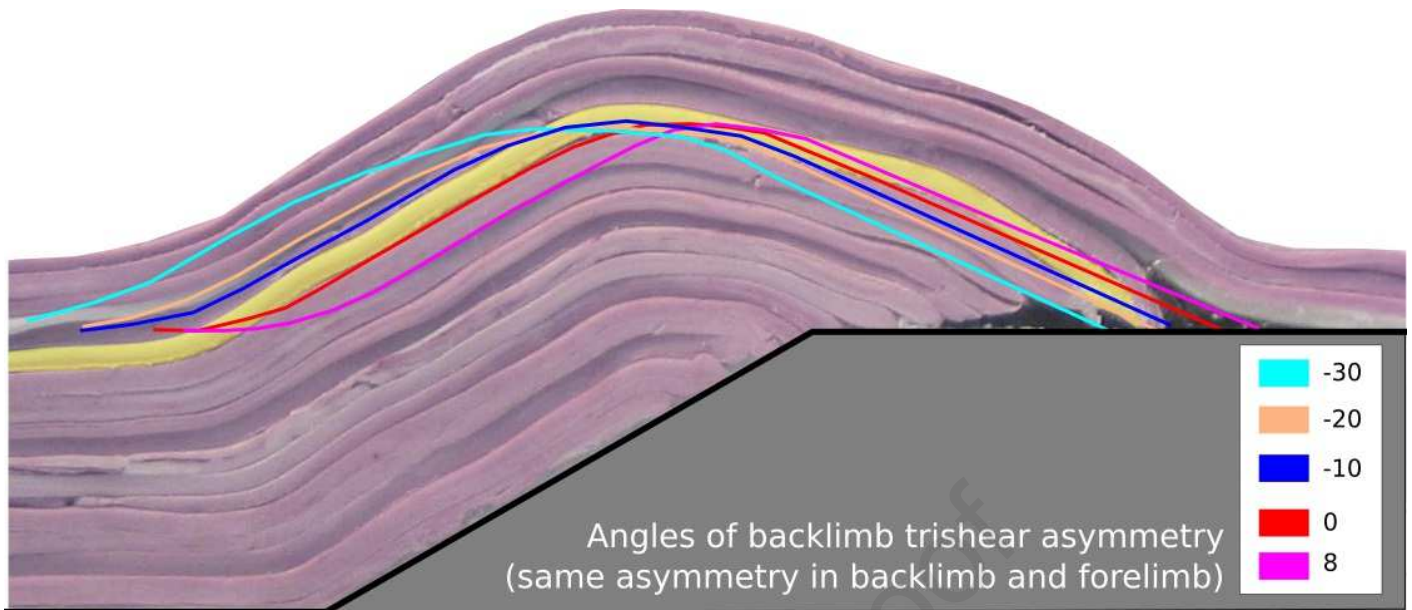


Figure 5: Curves obtained from applying backlimb trishear, using 30° as apical angle and different asymmetries (see figure inset color code). Comparing with the fold shape of the analog model (using the yellow highlighted bed as reference), -10° asymmetry works better for backlimb while $+8^\circ$ asymmetry works better for forelimb.

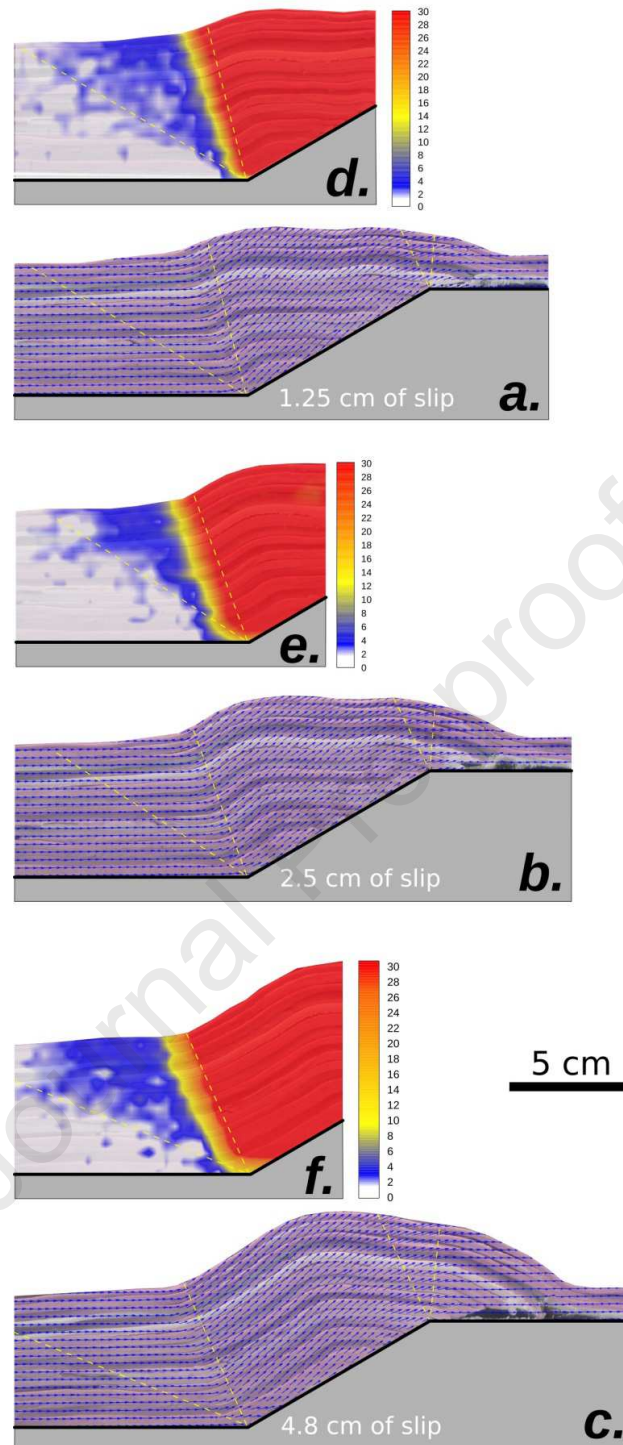


Figure 6: Total component of the displacement field represented with blue vectors. **a.** Photograph of the experiment with 1.25 cm of applied slip. **b.** Photograph of the experiment with 2.5 cm of applied slip. **c.** Photograph of the experiment with 4.8 cm of applied slip. **d., e., and f.** Color map graphics of slip vectors direction measured anticlockwise from the x-axis. Yellow dashed lines represent the backlimb trishear zones adjusted to the analog model.

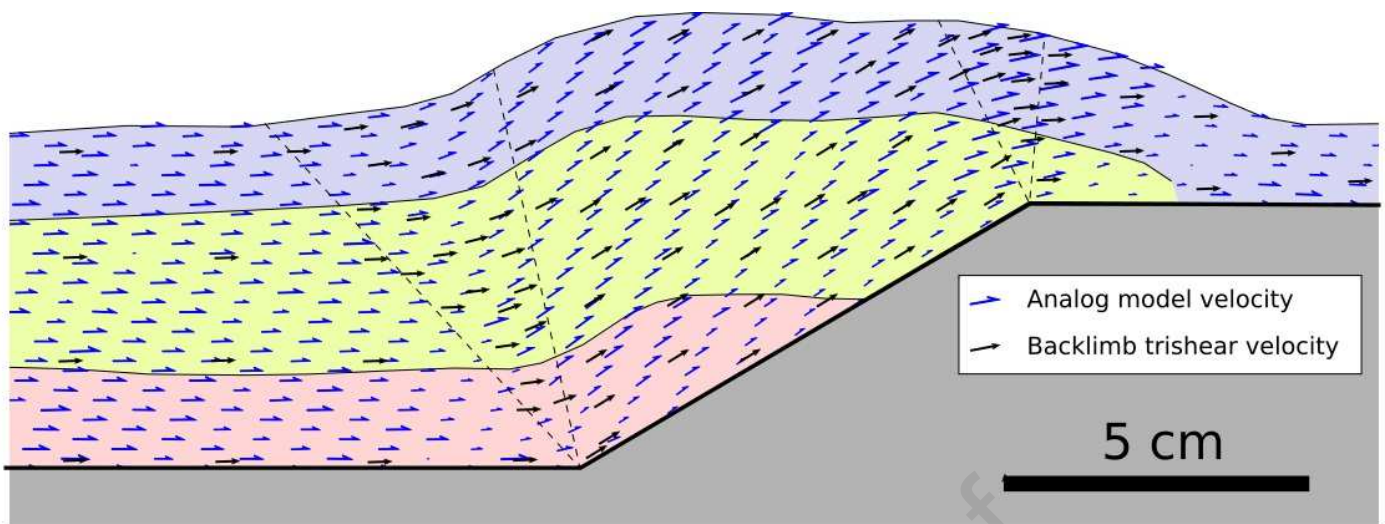


Figure 7: Comparison between displacement vectors obtained from the PIV analysis (blue arrows) performed from the serial images of the analog model and those obtained after applying backlimb kinematic model (black arrows) using -10° and $+8^\circ$ asymmetries for backlimb and forelimb, respectively, and an apical angle of 30° . The black dashed lines represent the trishear zones for the backlimb trishear model. Total component of the displacement field represented with blue vectors corresponds to 2.5 cm of applied slip.

a.

Kinematic model	A (cm/h)	B (cm/h)	C (cm/h)
CFBF	6.8	6.8	3.9
FPF	6.8	6.8	6.8
ISh (vert.)	6.8	7.85	6.8
CH	6.8	6.8	3.9
BLT (-10;+8)	6.8	6.19	5.7

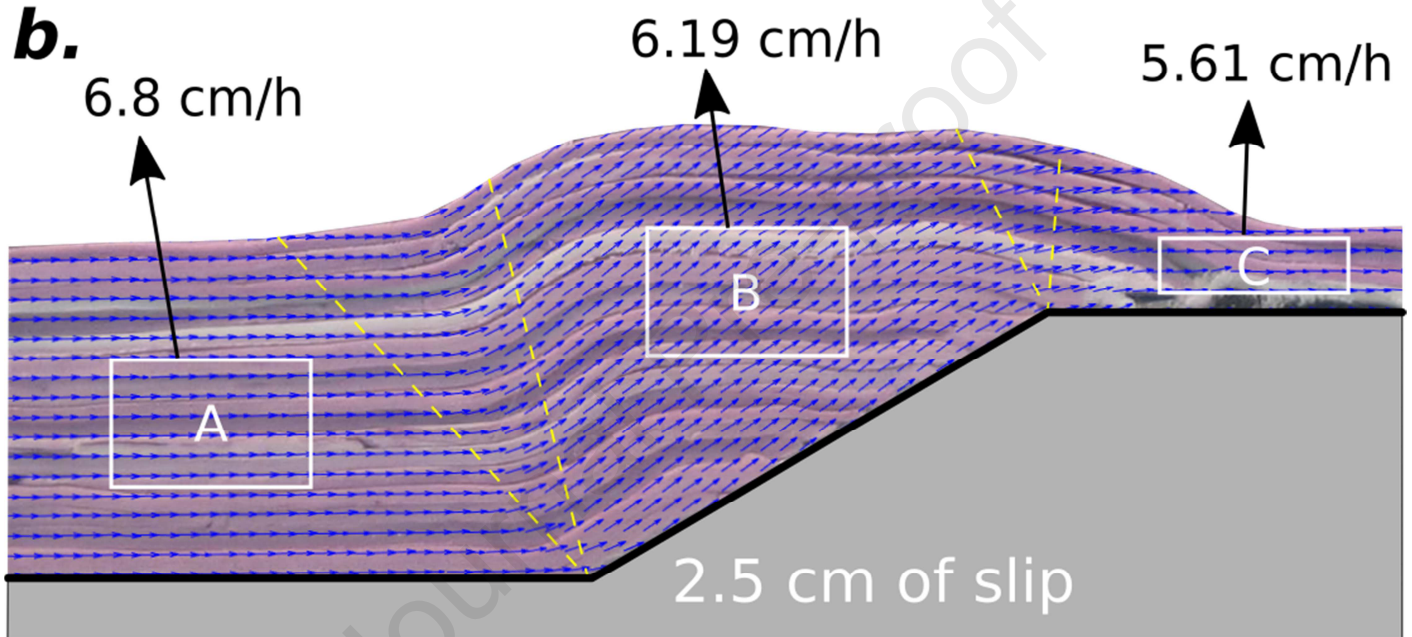
b.

Figure 8: a. Table presenting different kinematic models and their velocities for the following regions: A. above basal plane. B. above plane over the ramp and C. above the upper plane. Kinematic models used are: classical fault bend folding (CFBF), fault parallel flow (FPF), vertical shear (ISh), curvilinear hinge model (CH) and backlimb trishear (BLT). **b.** Photograph of the experiment with 2.5 cm of applied slip and the total component of the displacement field represented with blue vectors. Yellow dashed lines represent the backlimb trishear zones. White rectangles illustrate the sectors from which the average velocity is calculated.

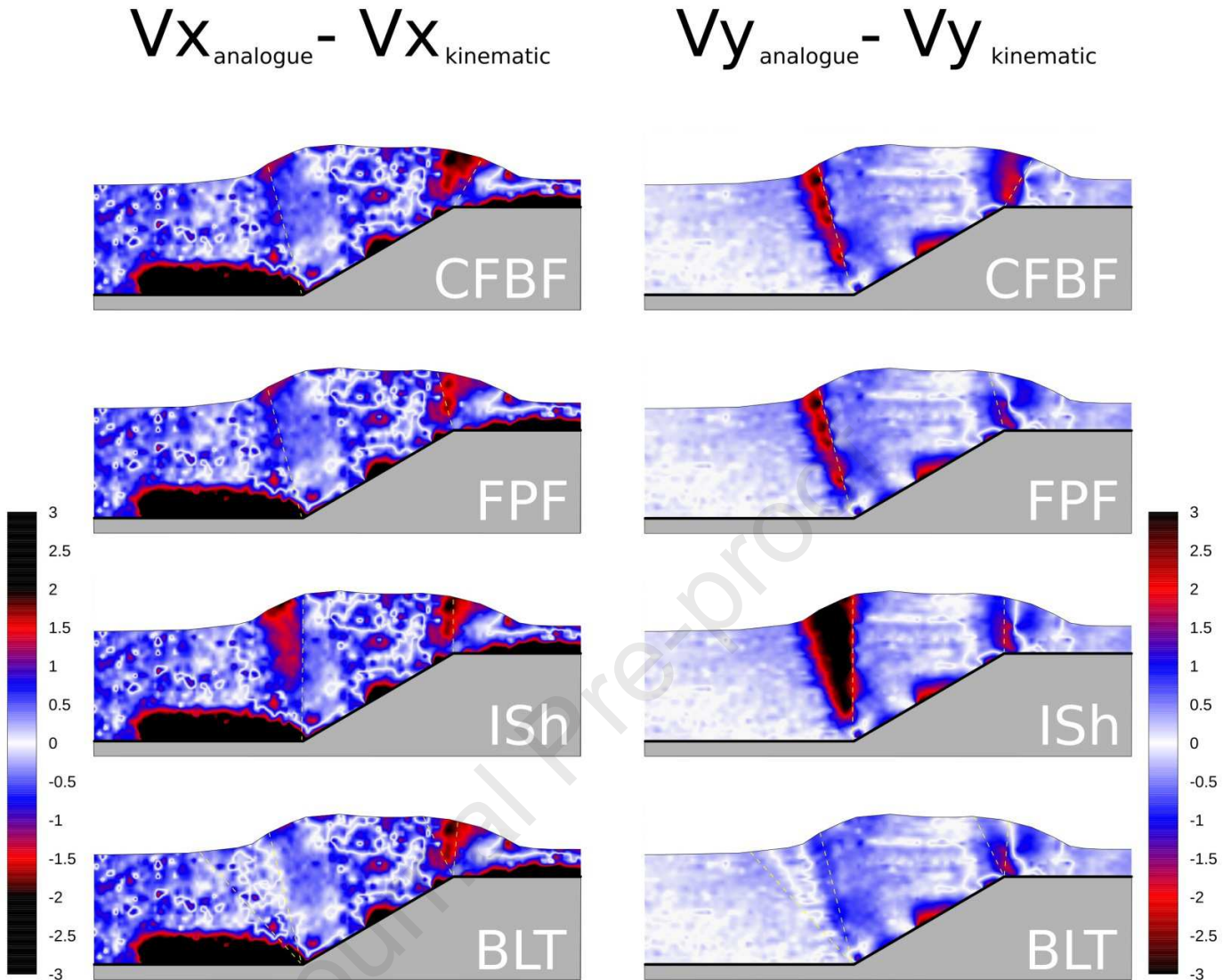


Figure 9: Resultant from the subtraction of kinematic models velocities to the analog model velocities. Kinematic models used are: classical fault bend folding (CFBF), fault parallel flow (FPF), vertical shear (ISh), and backlimb trishear (BLT) using -10° and $+8^\circ$ asymmetries for backlimb and forelimb, respectively, and an apical angle of 30° . The left column shows the subtraction of horizontal velocity components (Vx) and the right column, the subtraction of vertical velocity components (Vy). Note that the BLT model is the one with the least differences with respect to the analog model.

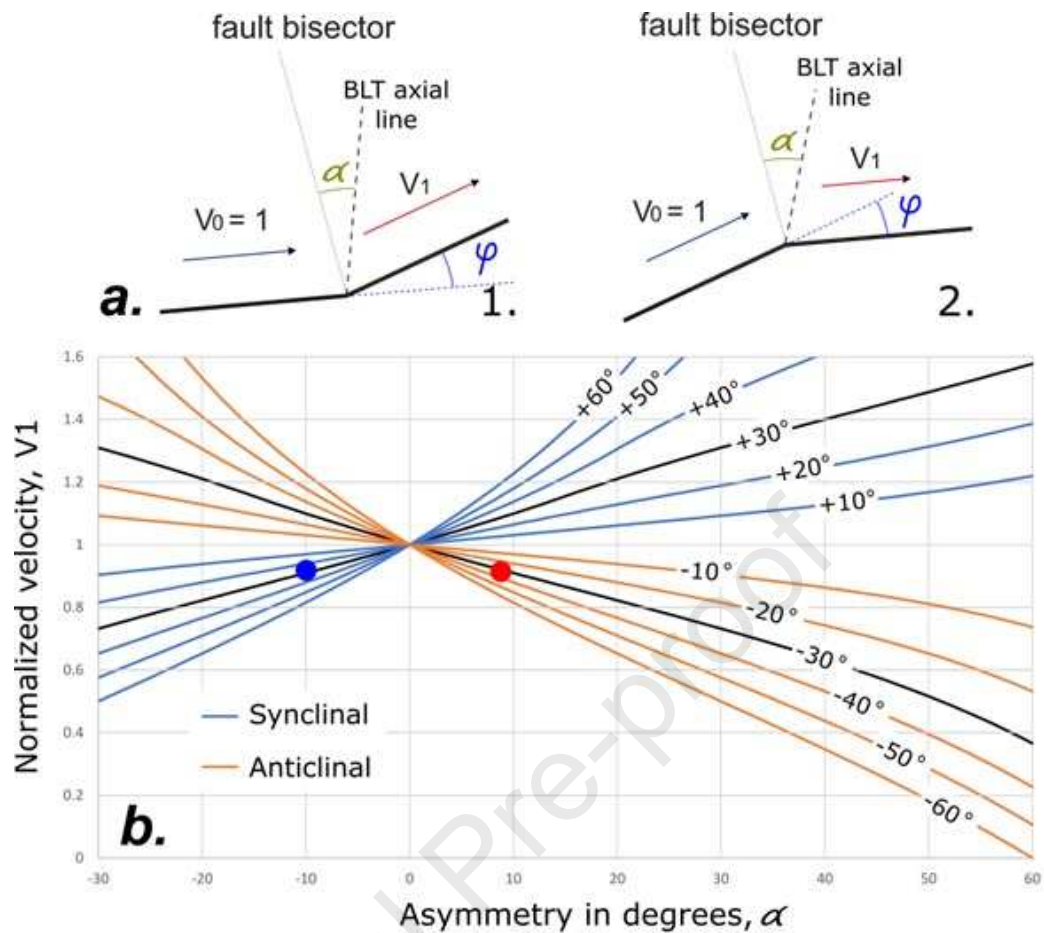


Figure 10: a. Scheme showing resultant velocity (V_1) after a fault bend, calculated with respect to a normalized to 1 input velocity (V_0), the angle between both sections of the fault (φ) and the asymmetry of the backlimb trishear zone (α). V_0 and V_1 are the velocities on either side of the triangular zone. φ is the angle between both sections of the fault. α is the angle of asymmetry of the backlimb trishear zone - the angle between the bisector of the fault bend angle (in a fine black straight line) and the bisector of the apical angle of backlimb trishear (BLT axial line, dashed). 1. Syncline bending of the fault (positives φ). 2. Anticlinal bending (negatives φ) (Modified from Cristallini and Allmendinger 2002). **b.** Curves showing resolution for Cristallini and Allmendinger (2002) velocity variation equation. Blue curves are for syncline bending of the fault (positives φ), while the orange curves are for anticlinal bendings (negatives φ). In the analog model, the angle of the fault was 30° (black curves). Blue point corresponds to $\alpha = -10^\circ$ asymmetry of the backlimb adjusted to the experiment fold (Figures 4e and 7). Red point corresponds to the $\alpha = +8^\circ$ asymmetry of the forelimb adjusted to the experiment fold (Figures 4e and 7).

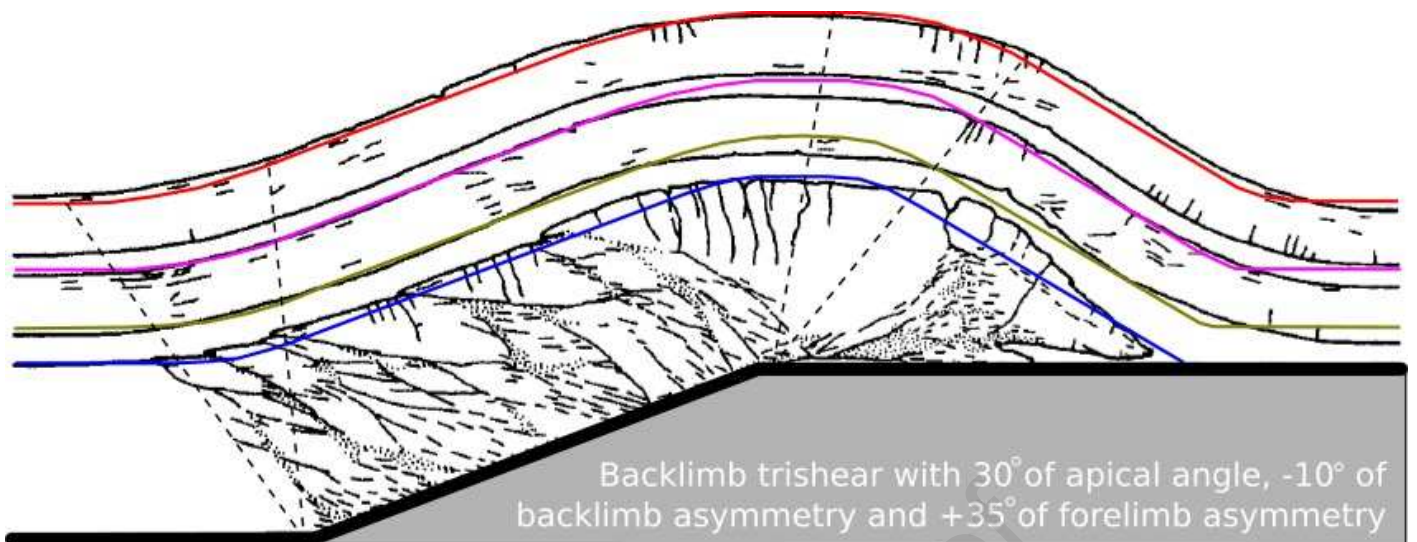
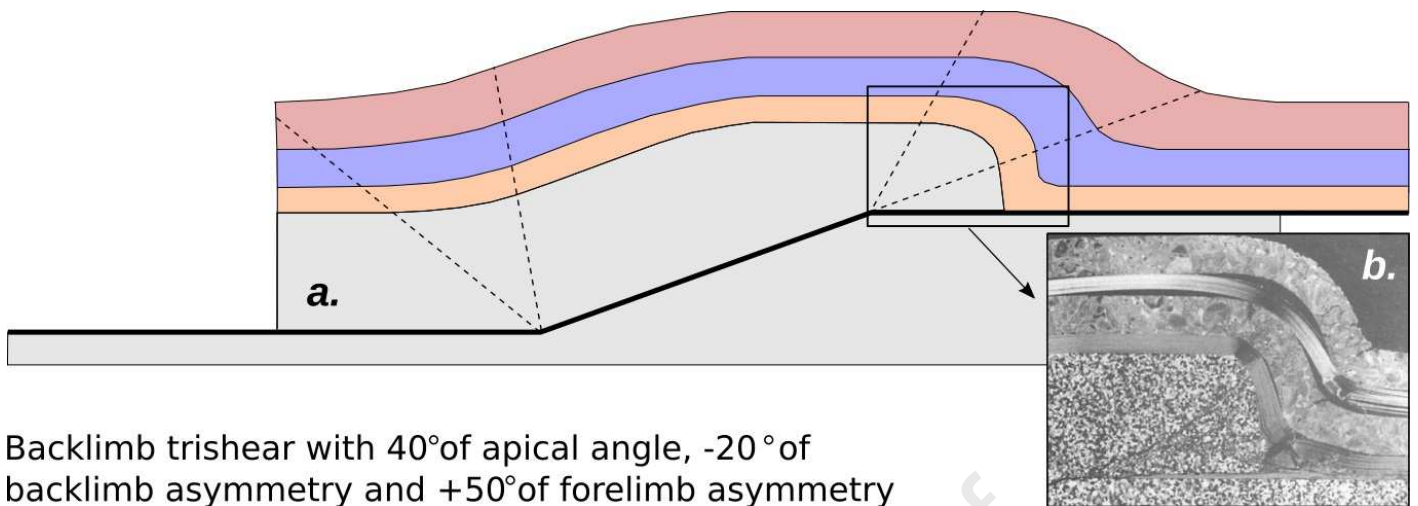


Figure 11: Comparison between analog model from Chester et al. 1991 and a backlimb trishear with 30° of apical angle, $\alpha = -10^\circ$ asymmetry for the backlimb and $\alpha = +35^\circ$ asymmetry for the forelimb.



Backlimb trishear with 40° of apical angle, -20° of backlimb asymmetry and $+50^\circ$ of forelimb asymmetry

Figure 12: Backlimb trishear model reconstruction using ANDINO 3D software. The geometry obtained in the analog model with a high dipping forelimb can be reproduced. **a.** Backlimb trishear model with 40° of apical angle, $\alpha = -20^\circ$ asymmetry for the backlimb and $\alpha = +50^\circ$ asymmetry for the forelimb. **b.** Analog model from Chester et al. 1991.

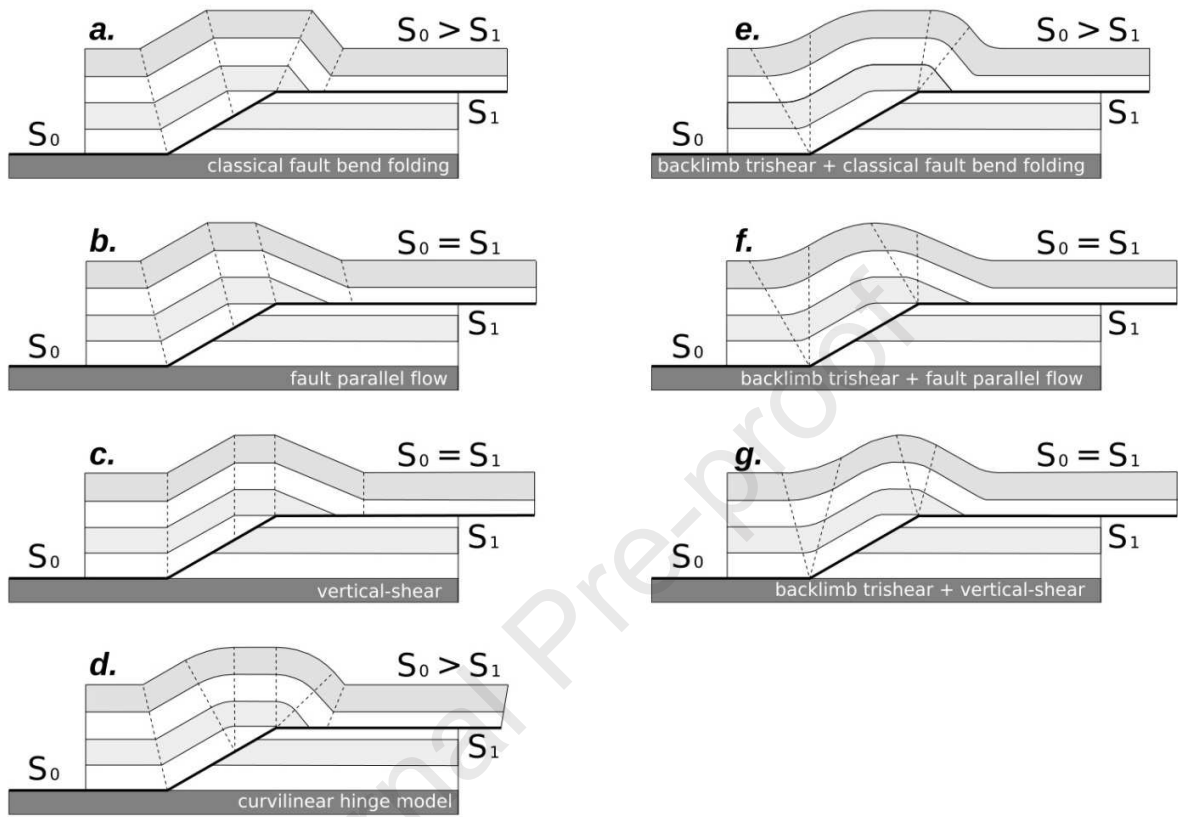
Black and white versions:**Figure 1**

Figure 3

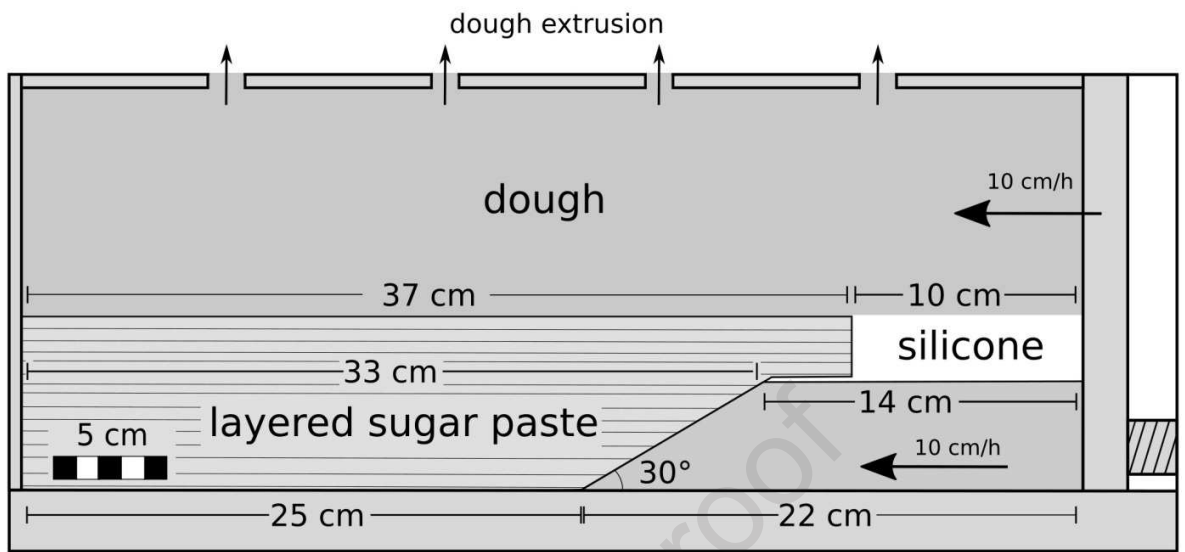


Figure 4

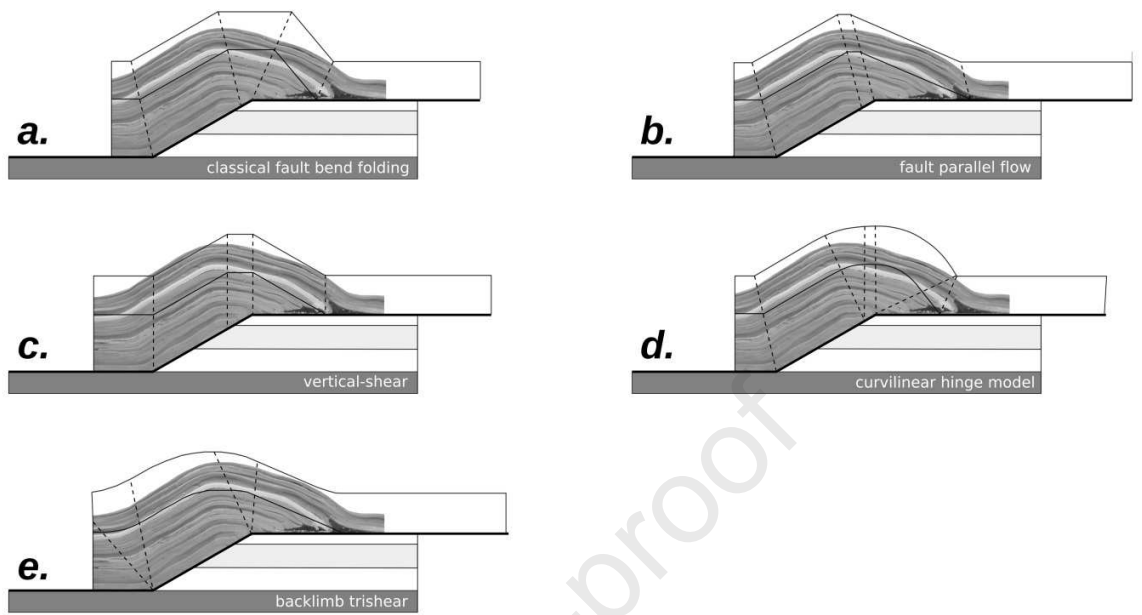


Figure 5

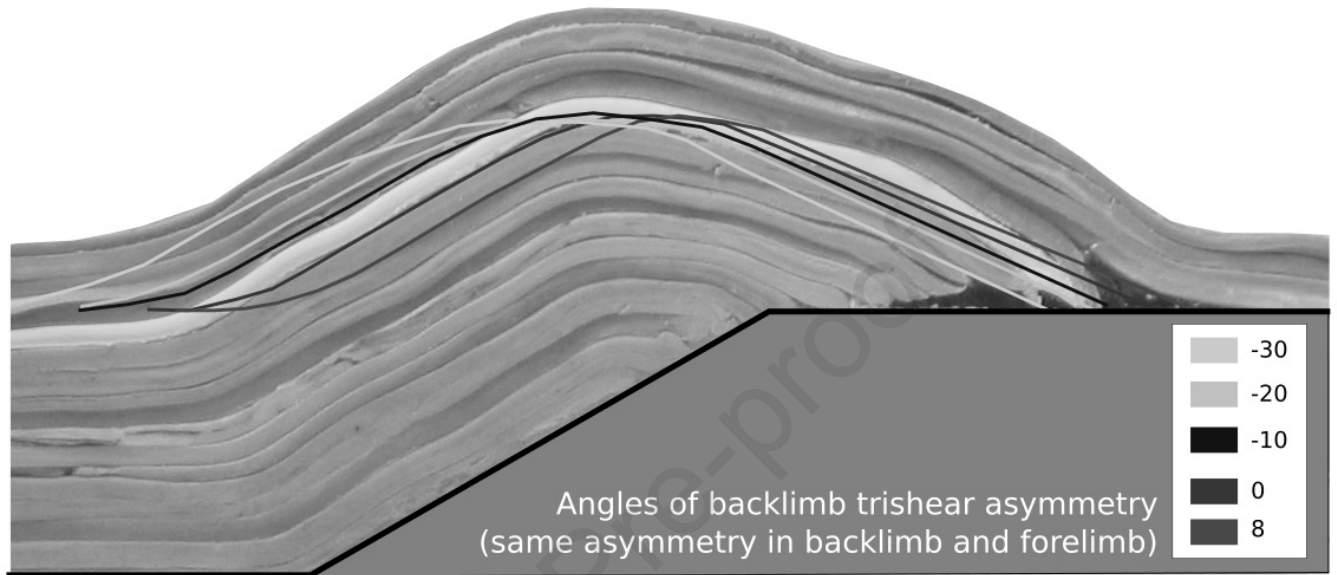


Figure 6

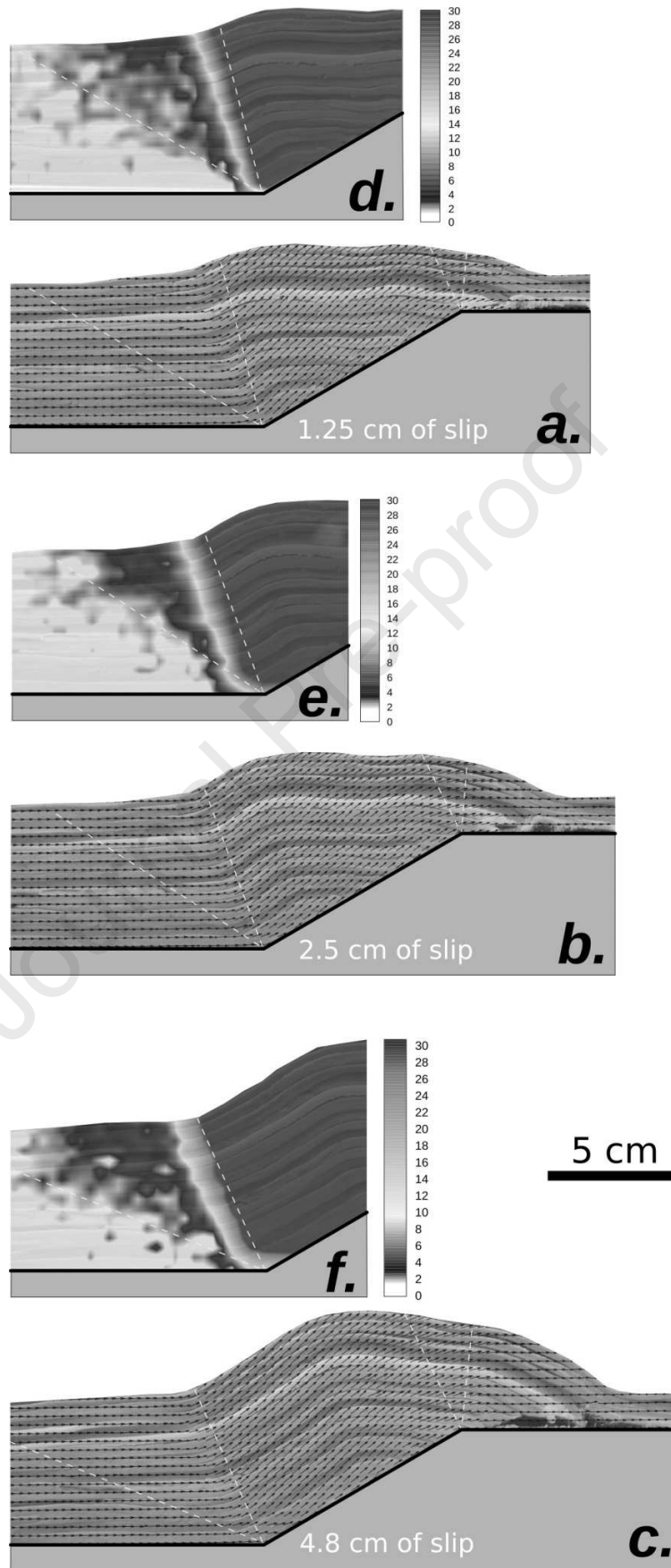


Figure 7

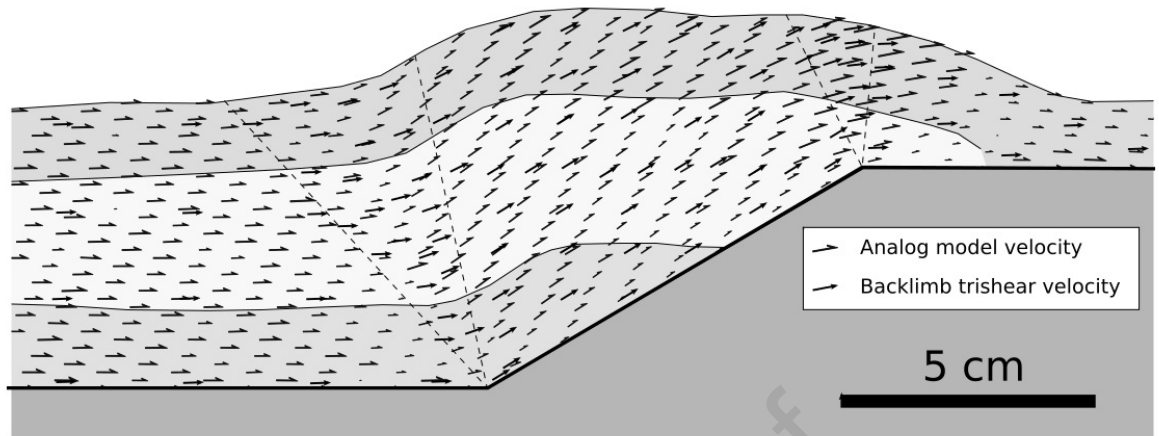


Figure 8

a.

Kinematic model	A (cm/h)	B (cm/h)	C (cm/h)
CFBF	6.8	6.8	3.9
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ISh (vert.)	6.8	7.85	6.8
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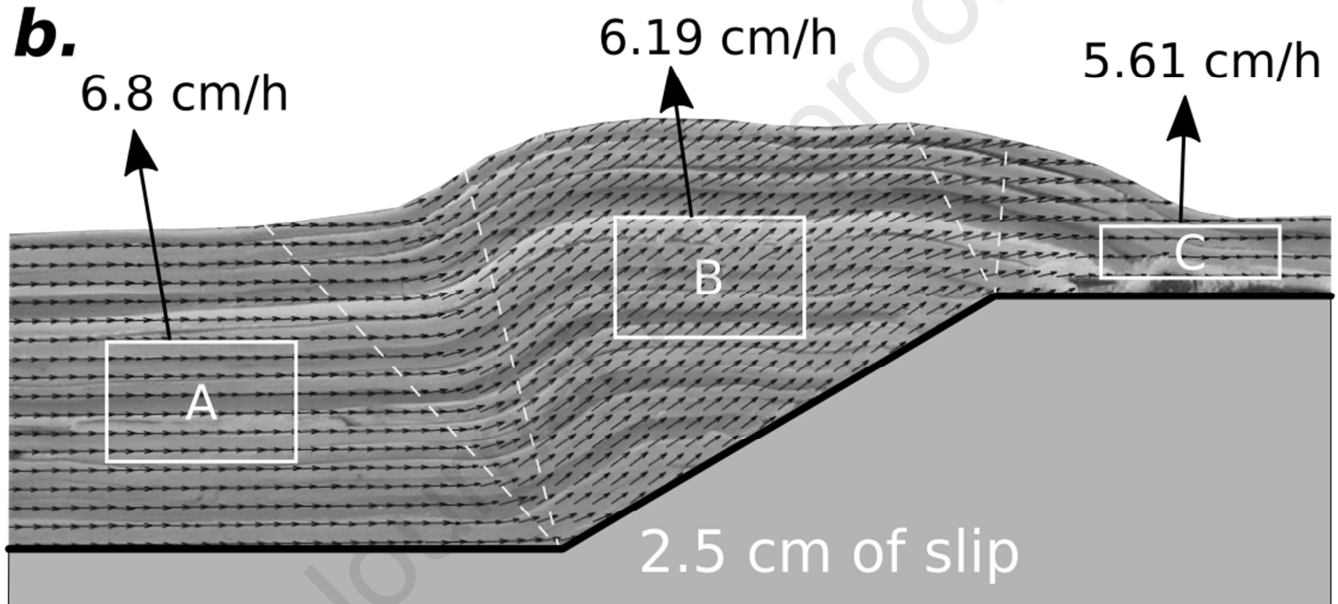
b.

Figure 9

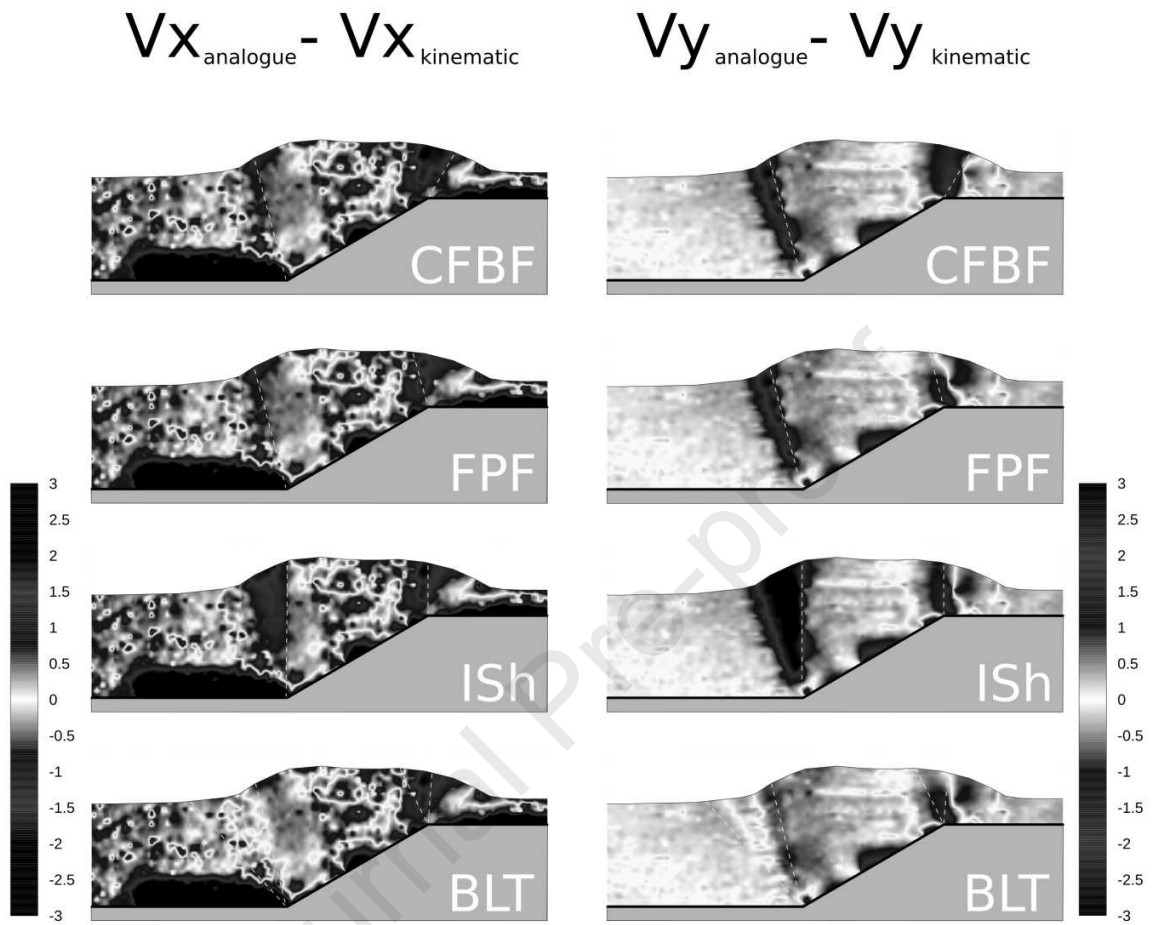


Figure 10

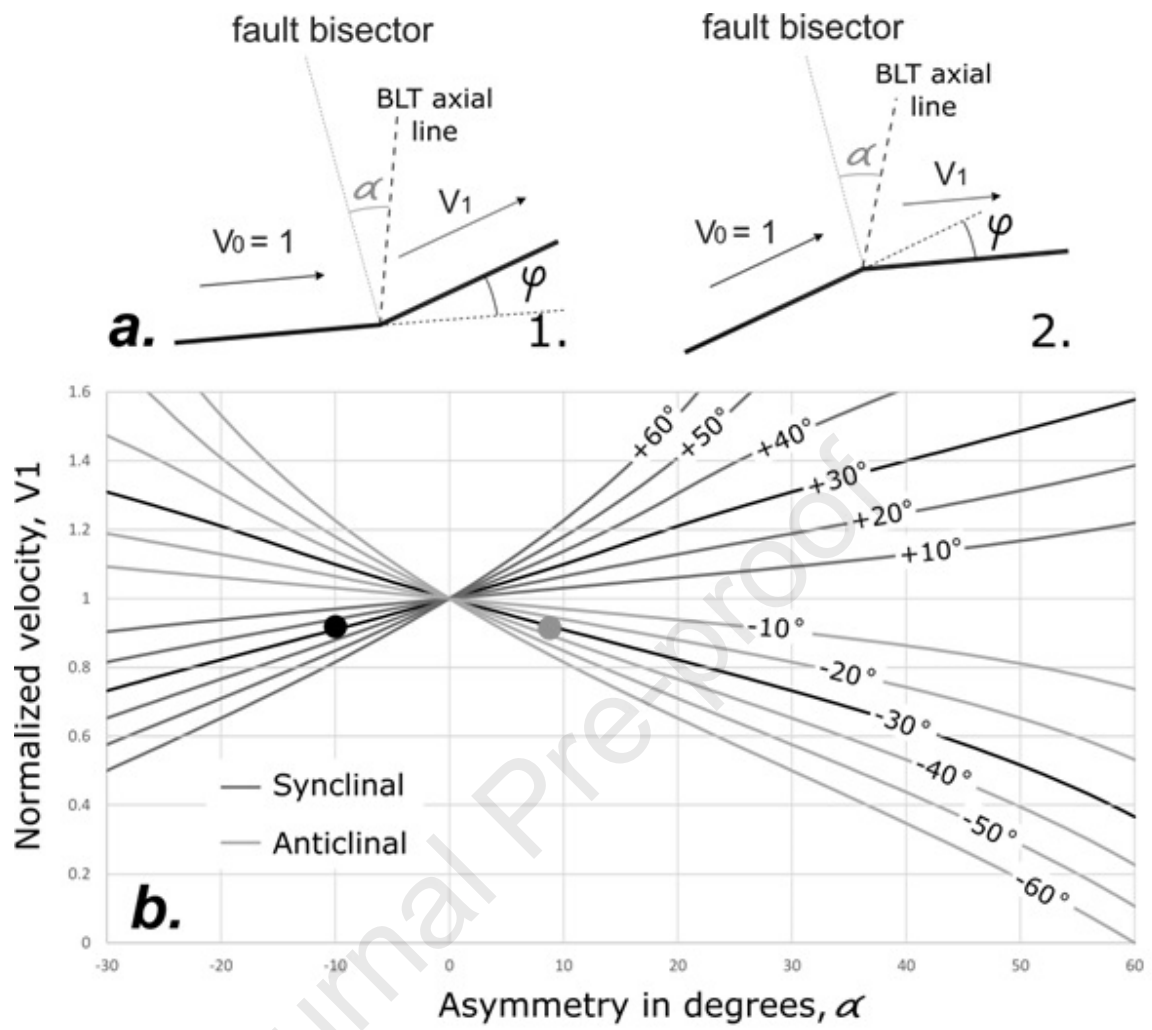


Figure 11

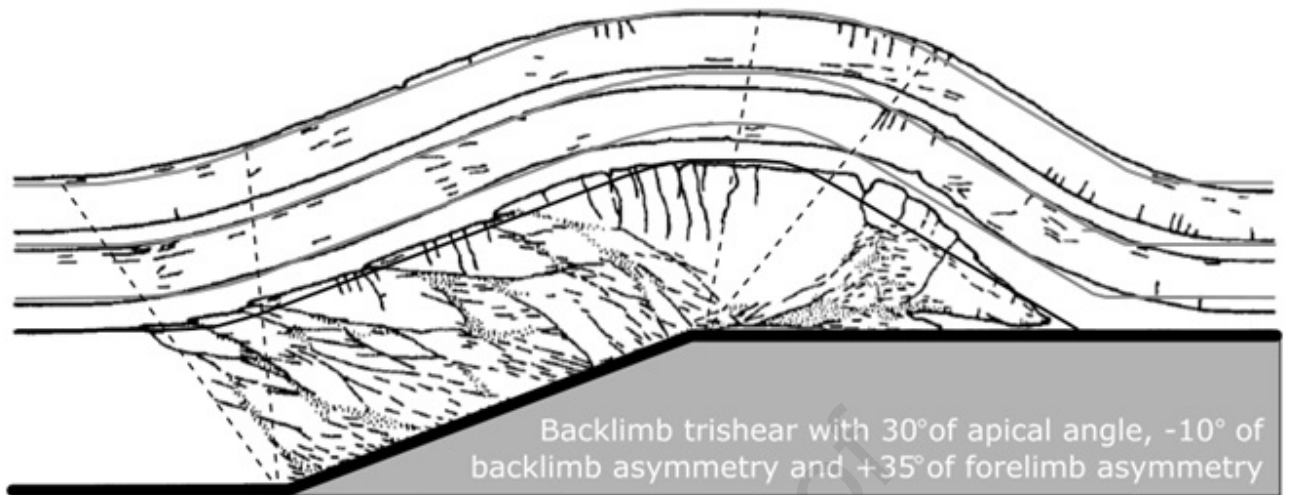
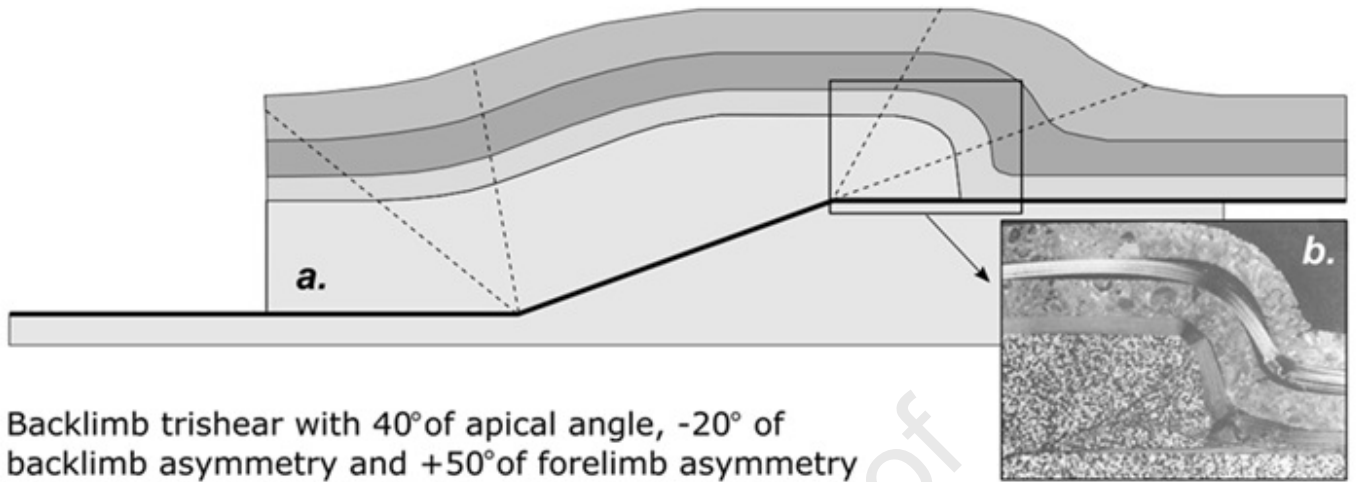


Figure 12



Highlights:

Analogue model was performed to obtain the velocity field during deformation.

Documentation of velocity vectors comes from particle image velocimetry.

Backlimb trishear can represent the geometry and directions of particles velocities.

Steeper-dipping forelimbs can be reproduced using high positive asymmetries.

Backlimb trishear apical angle can be used to control the shape of the hinges.

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Declaration of interests

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

The authors declare the following financial interests/personal relationships which may be considered as potential competing interests:

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