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THE EFFECT OF VIRTUAL MASS
ON THE CHARACTERISTICS AND THE NUMERICAL
STABILITY IN TWO-PHASE FLOWS

by

H. C. No and M. S. Kazimi

Energy Laboratory Report No. MIT-EL 81-023

April 1981



Energy Laboratory
and
Department of Nuclear Engineering

Massachusetts Institute of Technology
Cambridge, Mass. 02139

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REPORTS IN REACTOR THERMAL HYDRAULICS RELATED TO THE
MIT ENERGY LABORATORY ELECTRIC POWER PROGRAM

A. Topical Reports (For availability check Energy Laboratory Headquarters,
Headquarters, Room E19-439, MIT, Cambridge,
Massachusetts 02139)

- A.1 General Applications
- A.2 PWR Applications
- A.3 BWR Applications
- A.4 LMFBR Applications

A.1 J.E. Kelly, J. Loomis, L. Wolf, "LWR Core Thermal-Hydraulic Analysis--
Assessment and Comparison of the Range of Applicability of the Codes
COBRA-IIIC/MIT and COBRA-IV-1," MIT Energy Laboratory Report No.
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M. S. Kazimi and M. Massoud, "A Condensed Review of Nuclear Reactor
Thermal-Hydraulic Computer Codes for Two-Phase Flow Analysis," MIT
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J.E. Kelly and M.S. Kazimi, "Development and Testing of the Three
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J.N. Loomis and W.D. Hinkle, "Reactor Core Thermal-Hydraulic Analysis--
Improvement and Application of the Code COBRA-IIIC/MIT," MIT Energy
Laboratory Report No. MIT-EL-80-027, September 1980.

D.P. Griggs, A.F. Henry and M.S. Kazimi, "Development of a Three-
Dimensional Two-Fluid Code with Transient Neutronic Feedback for LWR
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J.E. Kelly, S.P. Kao and M.S. Kazimi, "THERMIT-2: A Two-Fluid Model
for Light Water Reactor Subchannel Transient Analysis," MIT Energy
Laboratory Report No. MIT-EL-81-014, April 1981.

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Characteristics and the Numerical Stability in Two-Phase Flows,"
MIT Energy Laboratory Report No. MIT-EL-81-023, April 1981.

J.W. Jackson and N.E. Todreas, "COBRA IIIC/MIT-2: A Digital Computer
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A Version of THERMIT for both Core-Wide and Subchannel Analysis of
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August 1981.

- A.2 P. Moreno, C. Chiu, R. Bowring, E. Khan, J. Liu, and N. Todreas, "Methods for Steady-State Thermal/Hydraulic Analysis of PWR Cores," MIT Energy Laboratory Report No. MIT-EL 76-006, Rev. 1, July 1977, (Orig. 3/77).

J. Liu and N. Todreas, "Transient Thermal Analysis of PWR's by a Single Pass Procedure Using a Simplified Model Layout," MIT Energy Laboratory Report No. MIT-EL 77-008, Final, February 1979 (Draft, June 1977).

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- A.3 L. Guillebaud, A. Levin, W. Boyd, A. Faya, and L. Wolf, "WOSUB-A Subchannel Code for Steady-State and Transient Thermal-Hydraulic Analysis of Boiling Water Reactor Fuel Bundles," Vol. II, Users Manual, MIT-EL 78-024, July 1977.

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B. Papers

- B.1 General Applications
- B.2 PWR Applications
- B.3 BWR Applications
- B.4 LMFBR Applications

- B.1 J.E. Kelly and M.S. Kazimi, "Development of the Two-Fluid Multi-Dimensional Code THERMIT for LWR Analysis," Heat Transfer-Orlando 1980, AIChE Symposium Series 199, Vol. 76, August 1980.

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- B.2 P. Moreno, J. Kiu, E. Khan, N. Todreas, "Steady State Thermal Analysis of PWR's by a Single Pass Procedure Using a Simplified Method," American Nuclear Society Transactions, Vol. 26.

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- B.3 L. Wolf and A. Faya, "A BWR Subchannel Code with Drift Flux and Vapor Diffusion Transport," American Nuclear Society Transactions, Vol. 28, 1978, p. 553.

S.P. Kao and M.S. Kazimi, "CHF Predictions In Rod Bundles," Trans. ANS, 35, 766 June 1981.

- B.4 W.D. Hinkle, (MIT), P.M. Tschamper (GE), M.H. Fontana (ORNL), R.E. Henry (ANL), and A. Padilla (HEDL), for U.S. Department of Energy, "LMFBR Safety & Sodium Boiling," paper presented at the ENS/ANS International Topical Meeting on Nuclear Reactor Safety, October 16-19, 1978, Brussels, Belgium.

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THE EFFECT OF VIRTUAL MASS ON THE CHARACTERISTICS
AND THE NUMERICAL STABILITY IN TWO-PHASE FLOWS

ABSTRACT

It is known that the typical six equation two-fluid model of the two-phase flow possesses complex characteristics, exhibits unbounded instabilities in the short-wavelength limit and constitutes an ill-posed initial value problem. Among the suggestions to overcome these difficulties, one model for the virtual mass force terms was studied here, because the virtual mass represents real physical effects to accomplish the dissipation for numerical stability. It was found that the virtual mass has a profound effect upon the mathematical characteristic and numerical stability. Here a quantitative bound on the coefficient of the virtual mass terms was suggested for mathematical hyperbolicity and numerical stability. It was concluded that the finite difference scheme with the virtual mass model is restricted only by the convective stability conditions with the above suggested value.

INTRODUCTION

Transient two-phase flow analysis is of importance in nuclear reactors under various accident conditions. Among the several models of two-phase flow the two-fluid model offers the most detailed and general description of two-phase flow. But it was reported in early work that the model has inherent instability problems. In 1965 Jarvis [1] tried to solve the two-phase equations using the two-fluid model for modeling the cooldown process in cryogenics and faced instability problems. He attributed the instabilities he encountered to the fact that the system was found to be nonhyperbolic. In 1967 Richtmyer and Morton [2] showed that, if the IVP is ill-posed, then no difference scheme that is consistent with the problem can be stable. In 1971 Siegmann encountered numerical instabilities in his transient sodium boiling code called MOC. In 1973 Boure appears to have encountered severe stability difficulties in the GEVATRAN code. In a round table discussion during the Fifth International Heat Transfer Conference [5], the instability problems in the two-fluid model were shown to be caused by an ill-posed problem. In 1976 Bryce [6] experienced large-scale pressure oscillations with RELAP-UK code. He demonstrated that simple two-fluid model with complex characteristics does not give solutions which converge as the mesh size and time step size are refined. He concluded that, though the solution may be obtained by using numerical technique, the significance of these solutions is not clear. In 1976 Lyczkowski [7] did a sample calculation illustrating error growth caused by complex characteristics. In the 1977 ANS topical meeting of thermal reactor safety, Anderson [8] reported his attempt to achieve stability by using virtual mass terms in his RISQUE code which was proposed

by Hilprecht. It was found that the virtual mass has a profound effect on the dynamics of two-phase flow. His code was used to perform numerical calculations on the behavior of interfacial waves. It was observed that for no virtual mass, the wave amplitude grows rapidly, but the computed result with virtual mass shows a stable oscillatory behavior for the wave amplitude. In 1979 Rivard and Travis [9] successfully dealt with critical flow with the two-fluid code, K-FIX. But for the results presented, the value of the interfacial drag function was chosen sufficiently large that there is no mean relative motion between the phases. In 1979 investigators at R.P.I. [10] showed that without virtual mass not only was it more costly to run the problem, but they could not even run the complete problem using their code, GEAR. Computer running time without virtual mass was forty times longer than that with virtual mass.

It may well be argued that equation sets with complex characteristics may still be adequate for a range of phenomena if the numerical method introduces sufficient dissipation to damp the high frequency instabilities. But obviously there are real physical effects to accomplish the dissipation needed for numerical stability. Among several candidates suggested for numerical stability, Cheng and Lahey's model [10] with the virtual mass terms will be studied here.

A. ILL-POSED PROBLEMS AND MATHEMATICAL STABILITY OF THE INITIAL VALUE PROBLEM (IVP)

Quantitative physical laws are idealizations of reality. As knowledge grows, we observe that a given physical situation can be idealized mathematically in a number of different ways. It is therefore important to characterize those reasonable ideal formulations. Hadamard [11] examined this problem, asserting that a physical formulation is well-posed if its solution exists, is unique, and depends continuously on the external (boundary) conditions. Existence and uniqueness are an affirmation of its principle of determination without which experiments could not be repeated with the expectation of consistent data. The continuous dependence criterion is an expression of the stability of the solution; a small change in any of the problem's data should produce only a correspondingly small change in the solution. The general one-dimensional problem can be represented in matrix form, as follows:

$$A(x) \frac{\partial x}{\partial t} + B(x) \frac{\partial x}{\partial z} + C(x) = 0, \quad (A.1)$$

where

x is a column vector of independent variables,
 $A(x)$ and $B(x)$ are the coefficient matrices, and
 $C(x)$ is a source or sink vector.

The IVP under consideration is to find a solution of Eq. (A.1) in some region

$$a \leq z \leq b,$$

subject to the initial condition

$$x(0,z) = u_0(z), \quad (\text{A.2})$$

and the value of x or its derivatives prescribed on the boundaries,

$$z = a \text{ and } z = b. \quad (\text{A.3})$$

When the definition of a well-posed problem by Hadamard is applied to the above system, the IVP is said to be well-posed if the solution of Eq. (A.1) exists, is unique, and depends continuously on both of the initial condition, Eq. (A.2) and the boundary conditions Eq. (A.3). Also system Eq. (A.1) is defined as hyperbolic if all values of the characteristics of Eq. (A.1) are distinct and real. According to Lax [12] the requirement of a well-posed problem in the linear partial differential equations with the form like Eq. (A.1) is the same as that of hyperbolicity. Along characteristic curves the highest-order derivatives in a PDE are indeterminate; characteristics may separate discontinuities in the solution. Therefore characteristics are trajectories of discontinuities in the solution. Physically it represents the travel of information in a physical system. The typical two-fluid models have four real characteristics (two convective velocities of liquid and vapor, and two relative sound velocities to both convective velocities of liquid and vapor through system) and two imaginary characteristics. The two imaginary characteristics may represent two relative velocities of interfacial wave propagation to both liquid and vapor with complex values which indicate an instability of interfacial waves.

Characteristics and stability are shown to be related only in the limit of large frequency. The characteristics, μ of Eq. (A.1) are defined by the equation

$$\text{Det} (\mu A - B) = 0 \quad (\text{A.4})$$

The local linear stability behavior of Eq. (A.1) is examined. x is replaced by $x_0 + \delta x$, and the result is linearized with respect to δx , resulting in

$$\begin{aligned} A(x_0) \frac{\partial(\delta x)}{\partial t} + B(x_0) \frac{\partial(\delta x)}{\partial z} + (\delta x) \left(\frac{\partial A}{\partial x} \right)_{x_0} \frac{\partial x_0}{\partial t} \\ + (\delta x) \left(\frac{\partial B}{\partial x} \right)_{x_0} \frac{\partial x_0}{\partial z} + \delta x \left(\frac{\partial C}{\partial x} \right)_{x_0} = 0 \end{aligned} \quad (\text{A.5})$$

As Eq. (A.5) describes small perturbations, δx about an unperturbed solution, a uniform steady-state unperturbed solution is assumed, which means that x_0 is independent of z and t . We take a wave form for the perturbed amount, δx

$$\delta x = \delta x_0 \exp[i(kz - \omega t)] \quad (\text{A.6})$$

Then Eq. (A.5) becomes

$$-i\omega A(x_0) \delta x_0 + ik B(x_0) \delta x_0 + \delta x_0 \left(\frac{\partial C}{\partial x} \right)_{x_0} = 0 \quad (\text{A.7})$$

Equation (A.7) is a homogeneous linear system in the components of δx . For a nontrivial solution the determinant of the coefficient matrix must vanish.

$$\text{Det}(-iwA + ikB + D) = 0, \quad (\text{A.8})$$

where

$$D = \left(\frac{\partial C}{\partial x} \right)_{x_0}^T, \quad (\text{A.9})$$

the superscript T denoting the matrix transpose. For nonzero k Eq. (A.8) can be rewritten as

$$\text{Det}\left(\frac{w}{k} A - B + \frac{i}{k} D\right) = 0 \quad (\text{A.10})$$

Inspection of Eq. (A.6) shows that the condition for stability is that $\text{Im}(w) \leq 0$ for all roots w . Comparison of Eq. (A.4) and Eq. (A.10) shows that, if $D=0$ or $k \rightarrow \infty$, the dispersion relations can be obtained from the characteristics simply by equating w/k to μ . Physically the instability at long wavelengths is the well-known Helmholtz instability. But the system which possesses complex characteristics exhibits unphysical and unbounded instabilities in the short-wavelength limit. One of the reasons why the basic system behaves like the above lies in the simplest choice for the interfacial pressure distribution in which it is constant and equal to the bulk pressure. Physically the above choice is consistent with the assumption that the two fluid velocities are equal. Therefore

this leads to an equation set possessing real characteristics only when the two fluid velocities are equal. This is the reason for the present study of the effect of the virtual mass here.

B. CHARACTERISTIC AND STABILITY ANALYSIS

The effect of virtual mass on the characteristics can be illustrated by adding the virtual mass terms to the one-dimensional momentum equations of both liquid and vapor. Here Cheng and Lahey's model for the virtual mass terms is considered. The conservation equations are as follows:

Conservation of vapor mass:

$$\frac{\partial}{\partial t} (\alpha_v \rho_v) + \frac{\partial}{\partial z} (\alpha_v \rho_v V_v) = \Gamma \quad (B.1)$$

Conservation of liquid mass:

$$\frac{\partial}{\partial t} (\alpha_l \rho_l) + \frac{\partial}{\partial z} (\alpha_l \rho_l V_l) = -\Gamma \quad (B.2)$$

Conservation of vapor momentum:

$$\begin{aligned} \alpha_v \rho_v \frac{\partial V_v}{\partial t} + \alpha_v \rho_v V_v \frac{\partial V_v}{\partial z} \\ = \Gamma (V_i - V_v) - \alpha_v \frac{\partial P}{\partial z} - \alpha_v [F_s + F_v] - \alpha_v \rho_v g - F_{wv}, \end{aligned} \quad (B.3)$$

where

V_i , F_s , F_v and F_{wv} are the interfacial velocity, the standard drag force per unit volume, the virtual mass force per unit volume, and the vapor wall friction force per unit volume, respectively.

$$F_s = k(V_v - V_l) \quad (B.4)$$

$$F_v = \rho_l C_v \left\{ \frac{\partial v_v}{\partial t} - \frac{\partial v_l}{\partial t} + v_v \frac{\partial}{\partial z} (v_v - v_l) \right. \\ \left. + (v_v - v_l) \left[(\lambda - 2) \frac{\partial v_v}{\partial z} + (1 - \lambda) \frac{\partial v_l}{\partial z} \right] \right\} \quad (B.5)$$

Conservation of liquid momentum

$$\alpha_l \rho_l \frac{\partial v_l}{\partial t} + \alpha_l \rho_l v_l \frac{\partial v_l}{\partial z} = \Gamma(v_l - v_i) - \alpha_l \frac{\partial P}{\partial z} \\ + \alpha_v [F_s + F_v] - \alpha_l \rho_l g - F_{wl}, \quad (B.6)$$

where F_{wl} is the liquid wall friction force per unit volume. For convenience by summing up Eq. (B.3) and Eq. (B.6), we can get the momentum equation

$$\alpha_v \rho_v \frac{\partial v_v}{\partial t} + \alpha_l \rho_l \frac{\partial v_l}{\partial t} + \alpha_v \rho_v v_v \frac{\partial v_v}{\partial z} + \alpha_l \rho_l v_l \frac{\partial v_l}{\partial z} \\ = \Gamma(v_l - v_v) - \frac{\partial P}{\partial z} - (\alpha_v \rho_v + \alpha_l \rho_l) g - (F_{wv} + F_{wl}) \quad (B.7)$$

By multiplying Eq. (B.3) by α_l and Eq. (B.6) by α_v and then subtracting each other, we obtain

$$\alpha_v (\alpha_l \rho_v + \rho_l C_v) \frac{\partial v_v}{\partial t} - \alpha_v (\alpha_l \rho_l + \rho_l C_v) \frac{\partial v_l}{\partial t} \\ + \{ [\alpha_l \alpha_v \rho_v + \alpha_v \rho_l C_v (\lambda - 1)] v_v - \alpha_v \rho_l C_v (\lambda - 2) v_l \} \frac{\partial v_v}{\partial z} \\ - \{ [\alpha_l \alpha_v \rho_l + \alpha_v \rho_l C_v (1 - \lambda)] v_l + \alpha_v \rho_l C_v \lambda v_v \} \frac{\partial v_l}{\partial z} \\ = \Gamma(v_i - \alpha_v v_l - \alpha_l v_v) - \alpha_v F_s + \alpha_l \alpha_v (\rho_l - \rho_v) g - \alpha_l F_{wv} + \alpha_v F_{wl} \quad (B.8)$$

Conservation of vapor energy:

$$\frac{\partial}{\partial t} (\alpha_v \rho_v e_v) + \frac{\partial}{\partial z} (\alpha_v \rho_v e_v V_v) + P \frac{\partial}{\partial z} (\alpha_v V_v) + P \frac{\partial \alpha_v}{\partial t} = Q_{wv} + Q_i, \quad (\text{B.9})$$

where Q_{wv} and Q_i are the heat transfer per unit volume from wall to vapor and from interface, respectively.

Conservation of liquid energy:

$$\frac{\partial}{\partial t} (\alpha_l \rho_l e_l) + \frac{\partial}{\partial z} (\alpha_l \rho_l e_l V_l) + P \frac{\partial}{\partial z} (\alpha_l V_l) - P \frac{\partial \alpha_v}{\partial t} = Q_{wl} - Q_i, \quad (\text{B.10})$$

where Q_{wl} is the heat transfer per unit volume from wall to liquid.

Let us construct the form of

$$A(x) \frac{\partial x}{\partial t} + B(x) \frac{\partial x}{\partial z} + C(x) = 0, \quad (\text{B.11})$$

where x is a column vector of independent variables,

$$x = (\alpha, P, V_v, V_l, e_v, e_l)^T$$

It is assumed that the density of both liquid and vapor is the function of pressure. Here a_v and a_l denote the sound velocity of vapor and liquid, respectively.

$$A = \begin{bmatrix} \rho_v & \alpha_v a_v^{-2} & & & & \\ -\rho_l & \alpha_l a_l^{-2} & & & & \\ & & \alpha_v \rho_v & \alpha_l \rho_l & & \\ & & \alpha_l \rho_v + \rho_l C_v & -\alpha_l \rho_l - \rho_l C_v & & \\ \rho_v e_v + P & \alpha_v e_v a_v^{-2} & & & \alpha_v \rho_v & \\ -\rho_l e_l - P & \alpha_l \rho_l a_l^{-2} & & & & \alpha_l \rho_l \end{bmatrix} \quad (B.12)$$

$$B = \begin{bmatrix} \rho_v V_v & \alpha_v V_v a_v^{-2} & \alpha_v \rho_v & & & \\ -\rho_l V_l & \alpha_l V_l a_l^{-2} & & \alpha_l \rho_l & & \\ & 1 & \alpha_v \rho_v V_v & \alpha_l \rho_l V_l & & \\ & & [\alpha_l \rho_v + \rho_l C_v (\lambda-1)] V_v & -[(1-\alpha) \rho_l + \rho_l C_v (1-\lambda)] V_l & & \\ & & -\rho_l C_v (\lambda-2) V_l & -\rho_l C_v V_v \lambda & & \\ \rho_v e_v V_v + P V_v & \alpha_v e_v V_v a_v^{-2} & \alpha_v \rho_v e_v + P \alpha_v & & \alpha_v \rho_v V_v & \\ -\rho_l e_l V_l - P V_l & (\alpha_l) e_l V_l a_l^{-2} & & \alpha_l \rho_l e_l - \alpha_v P & & \alpha_l \rho_l V_l \end{bmatrix} \quad (B.13)$$

In order to get the characteristics from Eq. (A.2)

$$\text{Det} (\mu A - B) = 0 \quad (B.14)$$

Reducing the terms related to the energy equations from Eq. (B.12), we get

$$\alpha_v \rho_v (\mu - V_v) \times \alpha_l \rho_l (\mu - V_l) \times$$

$$\text{Det} \begin{bmatrix} \rho_v (\mu - V_v) & \alpha_v a_v^{-2} (\mu - V_v) & -\alpha_v \rho_v & 0 \\ -\rho_l (\mu - V_l) & \alpha_l a_l^{-2} (\mu - V_l) & 0 & -\alpha_l \rho_l \\ 0 & -1 & \alpha_v \rho_v (\mu - V_v) & \alpha_l \rho_l (\mu - V_l) \\ 0 & 0 & (\alpha_l \rho_v + \rho_l C_v) (\mu - V_v) - (\alpha_l \rho_l + \rho_l C_v) (\mu - V_l) & -\rho_l C_v (\lambda - 2) (V_v - V_l) + \rho_l C_v \lambda (V_v - V_l) \end{bmatrix} = 0 \quad (\text{B.15})$$

From the terms related to the energy equations we obtained $\mu = V_v, V_l$. Physically this means that energy is transferred by only convection. For simplicity with the assumption that $a_v \gg V_v$ and $a_l \gg V_l$, that is, incompressible flow is used, we neglect

$$\alpha_l a_l^{-2} (\mu - V_l) \text{ and } \alpha_v a_v^{-2} (\mu - V_v)$$

We may predict that these terms are related to the sonic velocities transferred through liquid and vapor.

Now we get the simplified determinant form

$$\text{Det} \begin{bmatrix} \rho_v (\mu - V_v) & -\alpha_v \rho_v & 0 \\ -\rho_l (\mu - V_l) & 0 & -\alpha_l \rho_l \\ 0 & (\alpha_l \rho_v + \rho_l C_v) (\mu - V_v) - (\alpha_l \rho_l + \rho_l C_v) (\mu - V_l) & -\rho_l C_v (\lambda - 2) (V_v - V_l) + \rho_l C_v \lambda (V_v - V_l) \end{bmatrix} = 0 \quad (\text{B.16})$$

From Eq. (B.16) we obtain an algebraic equations of the form

$$aq^2 + bq + c = 0, \quad (\text{B.17})$$

where

$$a = \alpha_\ell^2 \rho_V + \alpha_V \alpha_\ell \rho_\ell + \gamma$$

$$b = [\gamma(2-\lambda) + 2\alpha_V \alpha_\ell \rho_\ell] V_r$$

$$c = [\alpha_V \alpha_\ell \rho_\ell + \alpha_V \gamma(1-\lambda)] V_r^2$$

$$\gamma = \rho_\ell C_V$$

$$q = \mu - V_V$$

$$V_r = V_V - V_\ell$$

In order for our system to have real and distinct characteristics

$$b^2 - 4ac > 0$$

$$\frac{b^2 - 4ac}{V_r^2} = [\gamma^2(\lambda^2 + 4(1-\lambda)\alpha_\ell) + 4\gamma(1-\lambda)\alpha_V \alpha_\ell^2(\rho_\ell - \rho_V) - 4\alpha_V \alpha_\ell^3 \rho_V \rho_\ell] \quad (\text{B.18})$$

i) There is no virtual mass; $C_V = 0$

$$b^2 - 4ac = -4\alpha_V \alpha_\ell^3 \rho_V \rho_\ell$$

Therefore the system with no virtual mass always has two complex characteristics except the single phase region

$$(\alpha_V = 0 \text{ or } \alpha_\ell = 0)$$

ii) If $\alpha_V \rightarrow 0$, $b^2 - 4ac = \gamma^2(\lambda - 2)^2$

If $C_V \neq 0$ and $\lambda \neq 2$, $\lim_{\alpha_V \rightarrow 0} (b^2 - 4ac) > 0$

iii) If $\alpha_\ell \rightarrow 0$, $b^2 - 4ac = \gamma^2 \lambda^2$

If $C_V \neq 0$ and $\lambda \neq 0$, $\lim_{\alpha_\ell \rightarrow 0} (b^2 - 4ac) > 0$

iv) If $\lambda = 1$, $b^2 - 4ac = \gamma^2 - 4\alpha_V \alpha_\ell^3 \rho_V \rho_\ell$

If $C_V > \sqrt{\frac{4 \alpha_V \alpha_\ell^3 \rho_V}{\rho_\ell}}$, $b^2 - 4ac > 0$ (B.19)

v) If $0 \leq \lambda < 1$, $C_V^2 [\lambda^2 + 4(1-\lambda)\alpha_\ell] + 4 \alpha_V \alpha_\ell^2 \gamma(1-\lambda)(1 - \frac{\rho_V}{\rho_\ell})$
 $> 4 \alpha_V \alpha_\ell^3 \frac{\rho_V}{\rho_\ell}$

As $4(1-\lambda)\alpha_\ell > 0$ and $4\alpha_V \alpha_\ell^2 \gamma(1-\lambda)(1 - \frac{\rho_V}{\rho_\ell}) > 0$,

$$C_V^2 \lambda^2 > 4\alpha_V \alpha_\ell^3 \frac{\rho_V}{\rho_\ell}$$

$$(C_V \lambda) > \sqrt{4\alpha_V \alpha_\ell^3 \frac{\rho_V}{\rho_\ell}}$$

vi) If $1 < \lambda < 2$, $C_V^2 (\lambda - 2)^2 - 4\alpha_V \alpha_\ell^2 C_V > 4\alpha_V \alpha_\ell^3 \frac{\rho_V}{\rho_\ell}$

$$C_V^2 (\lambda - 2)^2 > 4\alpha_V \alpha_\ell^2 [C_V + \alpha_\ell \frac{\rho_V}{\rho_\ell}]$$

Note that as λ approaches 2 or 0, the required value of C_V becomes larger and larger.

The effect of virtual mass on stability can be simply illustrated by using the characteristics analysis. If the mass exchange, standard drag force, wall friction, and the energy equations are neglected in order to know clearly the effect of virtual mass on stability, D in Eq. (A.10) reduces to zero.

Now as stated before, the dispersion relationship can be obtained from the characteristics simply by equating $\frac{w}{k}$ to μ .

$$w = k\mu \quad (\text{B.20})$$

The stability condition, $\text{Im}(w) \leq 0$ for all roots w is the same as the condition for the real and distinct characteristics.

C. NUMERICAL STABILITY

Here we shall examine a difference scheme formed in a similar way to THERMIT [13]. For the same reason as in the stability analysis, mass exchange, standard drag force, and wall friction are neglected. Also energy equations are dropped because they only represent convective properties as we say in the characteristic analysis

vapor mass

$$\frac{(\alpha_{v\rho_v})_j^{n+1} - (\alpha_{v\rho_v})_j^n}{\Delta t} + \frac{1}{\Delta z} [(\alpha_{v\rho_v}^n v_v^{n+1})_{j+1/2} - (\alpha_{v\rho_v}^n v_v^{n+1})_{j-1/2}] = 0 \quad (C.1)$$

liquid mass

$$\frac{(\alpha_{l\rho_l})_j^{n+1} - (\alpha_{l\rho_l})_j^n}{\Delta t} + \frac{1}{\Delta z} [(\alpha_{l\rho_l}^n v_l^{n+1})_{j+1/2} - (\alpha_{l\rho_l}^n v_l^{n+1})_{j-1/2}] = 0 \quad (C.2)$$

momentum equations

$$\begin{aligned} & (\alpha_{v\rho_v})^n \frac{(v_v^{n+1} - v_v^n)_{j+1/2}}{\Delta t} + (\alpha_{l\rho_l})^n \frac{(v_l^{n+1} - v_l^n)_{j+1/2}}{\Delta t} + (\alpha_{v\rho_v} v_v \frac{\Delta v_v}{\Delta z})_{j+1/2}^n \\ & + (\alpha_{l\rho_l} v_l \frac{\Delta v_l}{\Delta z})_{j+1/2}^n + \frac{(P_{j+1} - P_j)^{n+1}}{\Delta z} = 0 \end{aligned} \quad (C.3)$$

$$\begin{aligned} & \alpha_v^n [\alpha_{l\rho_v} + \rho_l C_v]^n \frac{(v_v^{n+1} - v_v^n)_{j+1/2}}{\Delta t} - \alpha_v^n [\alpha_{l\rho_l} + \rho_l C_v]^n \frac{(v_l^{n+1} - v_l^n)_{j+1/2}}{\Delta t} \\ & + \{ \langle \alpha_{l\rho_v} + \alpha_{v\rho_l} C_v (\lambda - 1) \rangle v_v - \alpha_{v\rho_l} C_v (\lambda - 2) v_l \} \frac{\Delta v_v}{\Delta z} \\ & - \{ \langle \alpha_{l\rho_v} + \alpha_{v\rho_l} C_v (1 - \lambda) \rangle v_l + \alpha_{v\rho_l} C_v \lambda v_v \} \frac{\Delta v_l}{\Delta z} \Big|_{j+1/2}^n = 0 \end{aligned} \quad (C.4)$$

The above numerical scheme can be modified as follows:

vapor mass

$$\begin{aligned} & \rho_{vj}^{n+1} (\alpha_v^{n+1} - \alpha_v^n)_j + \alpha_{vj}^n (\rho_v^{n+1} - \rho_v^n)_j + \frac{\Delta t}{\Delta z} [\rho_{vj}^n v_{vj+1/2}^{n+1} \\ & (\alpha_{vj}^n - \alpha_{vj-1}^n) + \alpha_{vj-1}^n v_{vj+1/2}^{n+1} (\rho_{vj}^n - \rho_{vj-1}^n) \\ & + \alpha_{vj-1}^n \rho_{vj-1}^n (v_{vj+1/2}^{n+1} - v_{vj-1/2}^{n+1})] = 0 \end{aligned} \quad (C.5)$$

liquid mass

Same as Eq. (C.5) with α_v replaced by α_ℓ , subscript v by ℓ . (C.6)

momentum equations

$$\begin{aligned} & (\alpha_v \rho_v)^n (v_v^{n+1} - v_v^n)_j + (\alpha_\ell \rho_\ell)^n (v_\ell^{n+1} - v_\ell^n)_j \\ & + \frac{\Delta t}{\Delta z} [(\alpha_v \rho_v v_v)^n (v_{vj} - v_{vj-1})^n + (\alpha_\ell \rho_\ell v_\ell)^n (v_{\ell j} - v_{\ell j-1})^n \\ & + (p_{j+1/2} - p_{j-1/2})^{n+1}] = 0 \end{aligned} \quad (C.7)$$

$$\begin{aligned} & [\alpha_\ell \rho_v + \rho_\ell C_v]^n (v_v^{n+1} - v_v^n)_j - [\alpha_\ell \rho_\ell + \rho_\ell C_v]^n (v_\ell^{n+1} - v_\ell^n)_j \\ & + \{ < [\alpha_\ell \rho_v + \rho_\ell C_v (\lambda-1)] v_v - \rho_\ell C_v (\lambda-2) v_\ell > \frac{\Delta t}{\Delta z} (v_{vj} - v_{vj-1}) \\ & - < [\alpha_\ell \rho_\ell + \rho_\ell C_v (1-\lambda)] v_\ell + \rho_\ell C_v \lambda v_v > \frac{\Delta t}{\Delta z} (v_{\ell j} - v_{\ell j-1}) > \}^n \\ & = 0 \end{aligned} \quad (C.8)$$

Treating the coefficients as constants and using Von Neumann local stability analysis, $U_j^n = \zeta^n \exp(ikj\Delta z)$, we obtain the determinant form for a nontrivial solution.

$$\text{Det} \begin{bmatrix} \rho_v(\zeta-1+\tilde{V}_v) & a_v^{-2}\alpha_v(\zeta-1+\tilde{V}_v) & \alpha_v\rho_v\zeta q & 0 \\ -\rho_\ell(\zeta-1+\tilde{V}_\ell) & a_\ell^{-2}\alpha_\ell(\zeta-1+\tilde{V}_\ell) & 0 & \alpha_\ell\rho_\ell\zeta q \\ 0 & \zeta q & \alpha_v\rho_v(\zeta-1+\tilde{V}_v) & \alpha_\ell\rho_\ell(\zeta-1+\tilde{V}_\ell) \\ 0 & 0 & (\alpha_\ell\rho_v+\rho_\ell C_v)(\zeta-1) + \{[\alpha_\ell\rho_v+\rho_\ell C_v(\zeta-1)]\tilde{V}_v - \rho_\ell C_v(\lambda-2)\tilde{V}_\ell\} & -(\alpha_\ell\rho_\ell+\rho_\ell C_v)(\zeta-1) - \{(\alpha_\ell\rho_\ell+\rho_\ell C_v(1-\lambda)]\tilde{V}_\ell + \rho_\ell C_v\lambda\tilde{V}_v\} \end{bmatrix} = 0 \quad (C.9)$$

where $\tilde{V}_v = \frac{\Delta t}{\Delta z} v_v(1-\exp(-ik\Delta z))$

$\tilde{V}_\ell = \frac{\Delta t}{\Delta z} v_\ell(1-\exp(-ik\Delta z))$

$q = 2i \frac{\Delta t}{\Delta z} \sin\left(\frac{k\Delta z}{2}\right)$

$k = \frac{\pi}{n\Delta z} : n=1 \sim J : J: \text{ the number of axial mesh}$

For simplicity we assume in the same way as in the previous analysis that $a_v \gg V_v, a_\ell \gg V_\ell$

Then the determinant reduces to the 3 x 3 determinant

$$\text{Det} \begin{bmatrix} \rho_v(\zeta-1+\tilde{V}_v) & \alpha_v\rho_v\zeta q & 0 \\ -\rho_\ell(\zeta-1+\tilde{V}_\ell) & 0 & \alpha_\ell\rho_\ell\zeta q \\ 0 & (\alpha_\ell\rho_v+\rho_\ell C_v)(\zeta-1+\tilde{V}_v) + \rho_\ell C_v(\lambda-2)(\tilde{V}_v-\tilde{V}_\ell) & -(\alpha_\ell\rho_\ell+\rho_\ell C_v)(\zeta-1+\tilde{V}_\ell) - \rho_\ell C_v\lambda(\tilde{V}_v-\tilde{V}_\ell) \end{bmatrix} = 0 \quad (C.10)$$

As we notice that ζq appears as a common coefficient, resolving the determinant we obtain the modified determinant

$$\text{Det} \begin{bmatrix} \rho_V(\mu - \tilde{V}_V) & -\alpha_V \rho_V & 0 \\ -\rho_\ell(\mu - \tilde{V}_\ell) & 0 & -\alpha_\ell \rho_\ell \\ 0 & (\alpha_\ell \rho_V + \rho_\ell C_V)(\mu - \tilde{V}_V) - \rho_\ell C_V(\lambda - 2)(\tilde{V}_V - \tilde{V}_\ell) & -(\alpha_\ell \rho_\ell + \rho_\ell C_V)(\mu - \tilde{V}_\ell) + \rho_\ell C_V \lambda (\tilde{V}_V - \tilde{V}_\ell) \end{bmatrix} = 0 \quad (\text{C.11})$$

where $\mu = -(\zeta - 1)$

The above determinant, Eq. (C.11) is in the exact same form as Eq. (B.16). Therefore we can use Eq. (B.17) as follows:

$$aq^2 + b\tilde{V}_r q + c\tilde{V}_r^2 = 0 \quad , \quad (\text{C.12})$$

where

$$a = \alpha_\ell^2 \rho_V + \alpha_V \alpha_\ell \rho_\ell + \rho_\ell C_V$$

$$b = \rho_\ell C_V (2 - \lambda) + 2\alpha_V \alpha_\ell \rho_\ell$$

$$c = \alpha_V \alpha_\ell \rho_\ell + \alpha_V \rho_\ell C_V (1 - \lambda)$$

$$q = \mu - \tilde{V}_V$$

$$\tilde{V}_r = \tilde{V}_V - \tilde{V}_\ell$$

For numerical stability the absolute value of the growth factor, $|\zeta|$, should be less than 1.

From Eq. (C.12)

$$q = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a} \tilde{V}_r \quad (\text{C.13})$$

$$\zeta - 1 = -\mu = -(\tilde{V}_V + q) \quad (C.14)$$

$$\zeta - 1 = - \left[\frac{(2a - b \pm \sqrt{b^2 - 4ac})}{2a} \tilde{V}_V + \frac{b \pm \sqrt{b^2 - 4ac}}{2a} \tilde{V}_\ell \right] \quad (C.15)$$

If $\lambda = 1$ and $\rho_\ell^2 C_V^2 \gg 4\alpha_V \alpha_\ell \rho_\ell \rho_V$, for $+\sqrt{b^2 - 4ac}$

$$\begin{aligned} \zeta - 1 &= - \frac{(\alpha_\ell^2 \rho_V + \rho_\ell C_V) \tilde{V}_V + (\alpha_V \alpha_\ell \rho_\ell) \tilde{V}_\ell}{\alpha_\ell^2 \rho_V + \alpha_V \alpha_\ell \rho_\ell + \rho_\ell C_V} \\ &= -p + p \exp(-ik\Delta z), \end{aligned} \quad (C.16)$$

where $p = \frac{(\alpha_\ell^2 \rho_V + \rho_\ell C_V) \frac{\Delta t}{\Delta z} V_V + (\alpha_V \alpha_\ell \rho_\ell) \frac{\Delta t}{\Delta z} V_\ell}{\alpha_\ell^2 \rho_V + \alpha_V \alpha_\ell \rho_\ell + \rho_\ell C_V}$

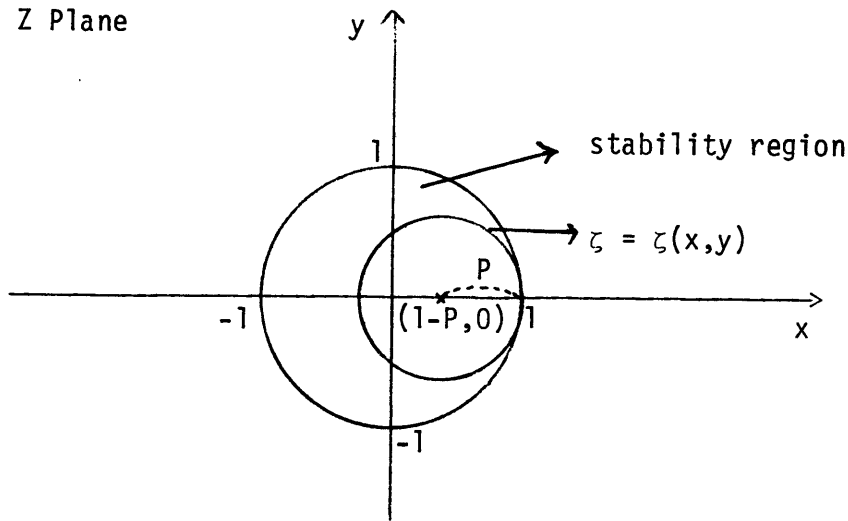


Fig. 1: Stability Diagram of Eq. (C.16)

For numerical stability $0 < P < 1$

$$\begin{aligned}
 0 &< (\alpha_{\ell}^2 \rho_v + \rho_{\ell} C_v) \frac{\Delta t}{\Delta z} V_v + (\alpha_v \alpha_{\ell} \rho_{\ell}) \frac{\Delta t}{\Delta z} V_{\ell} \\
 &< (\alpha_{\ell}^2 \rho_v + \rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell})
 \end{aligned} \tag{C.17}$$

If $\frac{\Delta t}{\Delta z} V_v < 1$ and $\frac{\Delta t}{\Delta z} V_{\ell} < 1$; the above inequality is always met.

For $-\sqrt{b^2 - 4ac}$

$$\begin{aligned}
 \zeta - 1 &= - \left[\frac{(\alpha_{\ell}^2 \rho_v) \frac{\Delta t}{\Delta z} \tilde{V}_v + (\rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell}) \frac{\Delta t}{\Delta z} \tilde{V}_{\ell}}{\alpha_{\ell}^2 \rho_v + \rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell}} \right] \\
 &= -r + r \exp(-ik\Delta z),
 \end{aligned} \tag{C.18}$$

$$\text{where } r = \frac{(\alpha_{\ell}^2 \rho_v) \frac{\Delta t}{\Delta z} V_v + (\rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell}) \frac{\Delta t}{\Delta z} V_{\ell}}{\alpha_{\ell}^2 \rho_v + \rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell}}$$

For numerical stability $0 < r < 1$

$$\begin{aligned}
 0 &< (\alpha_{\ell}^2 \rho_v) \frac{\Delta t}{\Delta z} V_v + (\rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell}) \frac{\Delta t}{\Delta z} V_{\ell} \\
 &< (\alpha_{\ell}^2 \rho_v + \rho_{\ell} C_v + \alpha_v \alpha_{\ell} \rho_{\ell})
 \end{aligned} \tag{C.19}$$

Also if $\frac{\Delta t}{\Delta z} V_v < 1$ and $\frac{\Delta t}{\Delta z} V_{\ell} < 1$, the above inequality is always met.

Therefore, it can be concluded that, if $\lambda = 1$ and

$C_v > \sqrt[3]{4\alpha_v \alpha_{\ell} \rho_v / \rho_{\ell}}$, the finite difference scheme is restricted only by the convective stability conditions.

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