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# Dynamics and mass transfer of rising bubbles in a homogenous swarm at large gas volume fraction 

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The present work focuses on the collective effect on both bubble dynamics and mass transfer in a dense homogeneous bubble swarm for gas volume fractions $\alpha$ up to $30 \%$. The experimental investigation is carried out with air bubbles rising in a square column filled with water. Bubble size and shape are determined by means of a high-speed camera equipped with a telecentric lens. Gas volume fraction and bubble velocity are measured by using a dual-tip optical probe. The combination of these two techniques allows us to determine the interfacial area between the gas and the liquid. The transfer of oxygen from the bubbles to the water is measured from the time evolution of the concentration of oxygen dissolved in water, which is obtained by means of the gassing-out method. Concerning the bubble dynamics, the average vertical velocity is observed to decrease with $\alpha$ in agreement with previous experimental and numerical investigations, while the bubble agitation turns out to be weakly dependent on $\alpha$. Concerning mass transfer, the Sherwood number is found to be very close to that of a single bubble rising at the same Reynolds number, provided the latter is based on the average vertical bubble velocity, which accounts for the effect of the gas volume fraction on the bubble rise velocity. This conclusion is valid for situations where the diffusion coefficient of the gas in the liquid is very low (high Péclet number) and the dissolved gas is well mixed at the scale of the bubble. It is understood by considering that the transfer occurs at the front part of the bubbles through a diffusion layer which is very thin compared with all flow length scales and where the flow remains similar to that of a single rising bubble.

Key words: bubble dynamics, drops and bubbles

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## 1. Introduction

Bubbly flows are usually employed in industry when the rate of mass transfer between a gas and a liquid is limited by the diffusion of the solute in the liquid. They combine the advantages of a large interfacial area by unit of volume and of an intense liquid agitation, which enhances the mixing of solute and accelerates chemical reactions. In many applications, the gas volume fraction $\alpha$ is larger than $20 \%$ and locally reaches much larger values. Bubbles can thus not be considered as isolated and collective effects have to be accounted for.

A first major collective effect is the decrease of the average bubble rise velocity $\left\langle V_{z}\right\rangle$ when increasing the gas volume fraction. The prediction of the increase of the bubble drag is an important issue for industrial applications. In the literature, several experimental works have investigated this phenomenon. Among them, the experiments of Wallis (1961), who investigated a homogenous bubble swarm of air bubbles in a soapy water solution, suggest that the rise velocity scales as $\left\langle V_{z}\right\rangle \propto(1-\alpha)$ up to a gas volume fraction of $30 \%$. This scaling law was established by considering the global conservation of the mass of gas, where the gas flow rate was measured directly and the gas volume fraction was deduced from the variation of the hydrostatic pressure. Using the same procedure and making an analogy with a fluidized bed, Bridge, Lapidus \& Elgin (1964) found a rather similar scaling, $\left\langle V_{z}\right\rangle \propto(1-\alpha)^{1.39}$, for the case of a countercurrent liquid flow, with air sparged into water, glycerine/water or water/isoamyl-alcohol mixtures, for $\alpha \leqslant 20 \%$. Wijngaarden \& Kapteijn (1990) determined the mean relative velocity of air bubbles in water by means of a technique based on electric conductance measurements and found that it scaled as ( $1-1.78 \alpha$ ) up to a gas volume fraction of $14 \%$. In the presence of a liquid flow Garnier, Lance \& Marié (2002) observed that $\left\langle V_{z}\right\rangle$ scaled as $\alpha^{1 / 3}$ for $\alpha \leqslant 40 \%$ by means of a dual-tip optical probe. For different various two-phase flow configurations, Ishii \& Chawla (1979) and Rusche \& Issa (2000) found more complex expressions. In order to estimate relative velocity in bubbly, droplet or particulate flows, Ishii \& Chawla (1979) proposed a model based on an effective viscosity of the two-phase mixture. Rusche \& Issa (2000) introduced a drag correction as a combination of a power law and an exponential function with coefficients that depend on the nature of the considered dispersed flow. Direct numerical simulations of a swarm of bubbles rising in a periodic domain have also been performed. For moderate Reynolds number $(\operatorname{Re}=O(10-100))$, using a front tracking method and avoiding bubble coalescence, the decrease of $\left\langle V_{z}\right\rangle$ with $\alpha$ has been confirmed for both spherical (Bunner \& Tryggvason 2002a,b) and ellipsoidal bubbles (Bunner \& Tryggvason 2003). Deformed bubbles at large Reynolds number ( $\operatorname{Re}=O(100-1000)$ ) for $\alpha \leqslant 45 \%$ have been simulated by Roghair et al. (2011) who used 20 Eulerian mesh points on the surface of each bubble. They observed that the decrease of the bubble velocity was affected by the bubble Eötvös number as well as by the value of the gas volume fraction. Despite the great number of experimental and numerical attempts, no general model for the rise velocity of bubbles exists yet, owing to the complexity of bubbly flows. Experimental investigations at large gas volume fractions ( $\alpha>15 \%$ ) with accurate determination of both the bubble geometry and velocity are thus still desirable.

A second collective effect of great significance is the modification of the interfacial rate of mass transfer when the gas volume fraction is increased. Despite the significant gas volume fractions that are present in most industrial applications, many studies make use of mass transfer models developed for isolated bubbles. These models are usually based on Higbie's penetration theory (Higbie 1935), but consider various definitions for the contact time: (i) the ratio of the bubble
diameter to the bubble rise velocity; (ii) the ratio of the bubble surface to the rate of surface formation (Nedeltchev, Jordan \& Schumpe 2006); or (iii) based on the eddy velocity for developed turbulent flows (Lamont \& Scott 1970; Kawase, Halard \& Moo-Young 1987; Linek et al. 2004). With a contact time defined as the ratio of the bubble diameter to the bubble rise velocity (i), the Higbie's penetration theory is also known as the Boussinesq solution (Boussinesq 1905). Numerical simulations (Takemura \& Yabe 1998; Figueroa \& Legendre 2010) have shown that this analytical solution appears to be very accurate at describing interfacial mass transfer for a single clean spherical bubble rising in a still liquid, at large bubble Reynolds and Péclet numbers. Moreover, the experiments by Alves, Vasconcelos \& Orvalho (2006) showed that this solution was still valid for the interfacial mass transfer of a single bubble fixed in a turbulent downward liquid flow, up to a certain dissipation rate of the turbulence. The Boussinesq solution has also been used as a closure law in Eulerian-Eulerian two-fluid simulations of industrial ozonation towers (Cockx et al. 1999) and aeration tanks for urban wastewater treatment (Fayolle et al. 2007) at low to moderate volume fractions ( $\alpha \leqslant 10 \%$ ). Higbie's penetration theory with a contact time based on the rate of surface formation (ii) has been found to provide a good estimate of the mass transfer rate in a pressurized bubble column for either water or organic liquids (Nedeltchev, Jordan \& Schumpe 2007). In the same time, Higbie's penetration theory with a contact time defined with eddy velocity (iii) has been preferred by Buffo, Vanni \& Marchisio (2012) and Petitti et al. (2013) to simulate gas-liquid mass transfer in stirred tank reactors.

As indicated above, Boussinesq solution is a priori limited to large bubble Reynolds and Péclet numbers and isolated spherical bubbles. Some corrections based on results for a single bubble have been introduced to account for the effect of finite Reynolds number (Darmana, Deen \& Kuipers 2005; Ayed, Chahed \& Roig 2007; Shimada, Tomiyama \& Ozaki 2007) and that of bubble deformation (Nedeltchev et al. 2007) in simulations of bubble columns. Such corrections are discussed by Takemura \& Yabe (1998) and Figueroa \& Legendre (2010). Reviews for mass transfer can be found in Clift, Grace \& Weber (1978) and in Michaelides (2006) for bubbles, but also for drops and particles. Most of these studies have focused on mass or heat transfer from a single inclusion. Their applicability in dense dispersed flow is an important issue.

In the last few decades, a few works have focused on collective effect upon mass transfer in a bubble swarm (Koynov \& Khinast 2005; Kishore, Chhabra \& Eswaran 2008; Colombet et al. 2011; Roghair 2012). Most of them are numerical works. Two-dimensional numerical simulations of mass transfer for different arrangements of bubbles have been performed by Koynov \& Khinast (2005) for small Reynolds numbers. For the case of three bubbles initially aligned horizontally, the authors observed a decrease of the Sherwood number. For this particular case, they noticed that, taking into account the reduced Reynolds number, the Sherwood number stays close to that of a single bubble. They also found a decrease of the Sherwood number for the case of bubbles which were initially aligned in the vertical direction. According to Koynov \& Khinast (2005), this is due to the fact that bubbles are rising in the wake of each other so that both the gradient of concentration and the interfacial mass flux are reduced. One of their conclusions is that 'Mass transfer in a bubble swarm depends both on the motion of the swarm as a whole and on the motion of the individual bubbles and, in general, does not follow trends observed in the single bubble cases'. For both Newtonian and non-Newtonian fluids, Kishore et al. (2008) used a 'cell model' of two concentric spheres to study numerically the collective effect of mass transfer for a clean spherical bubble. In that simplified approach, the
increase of gas volume fraction is modelled by a decrease of the bounding sphere. The results seem to suggest an increase of the Sherwood number with the increase of the gas volume fraction.

The effect of increasing the gas volume fraction on the gas-liquid mass transfer coefficient has been experimentally investigated by Colombet et al. (2011) for air bubbles in water. Thanks to a high-speed camera with a fixed focal lens, a particle tracking velocimetry (PTV) method was able to measure bubble volumes, shapes and velocities for gas volume fractions from 0.45 to $16.5 \%$. In this range, the mass transfer coefficient is found very close to that of a single bubble provided that the Reynolds number is based on the mean equivalent diameter and the average rising velocity of a bubble in the swarm, which suggests a weak influence of the collective effect on the mass transfer at high Péclet number. In a recent study using direct numerical simulation, Roghair (2012) found a marginal increase of the mass transfer coefficient $k_{L}$ with the increase of the gas volume fraction for 4 mm air bubbles rising in water at $R e \leqslant 1070, S c=1$ and $4 \leqslant \alpha \leqslant 40 \%$.

The objective of the present study is to investigate collective effect on the bubble dynamics and mass transfer in very dense homogeneous bubbly flows with controlled hydrodynamic conditions. For this purpose, accurate measurements of interfacial area, bubble diameter, deformation and rising velocity are first performed for $12.1 \leqslant \alpha \leqslant 33.9 \%$. Then, oxygen mass transfer experiments are conducted for $0.7 \leqslant \alpha \leqslant 29.6 \%$. The paper is organized as follows. Section 2 describes the experimental methods. Section 3 presents the dynamics of the bubbles while §4 shows the results concerning mass transfer. Section 5 is devoted to the analysis and the discussion of the results. Section 6 summarizes the main conclusions.

## 2. Experimental set-up and instrumentation

### 2.1. General description

The experimental set-up is described in figure $1(a)$. It has been used previously by Riboux, Risso \& Legendre (2010) and Colombet et al. (2011). Bubbles are injected through stainless steel capillaries (1) in a square glass column of $15 \mathrm{~cm} \times 15 \mathrm{~cm}$ cross-section and 100 cm high. The gas line is equipped with three different rotameters (2) and one manometer (3) to deal with a large range of gas flow rates and volume fractions. A three-way valve enables the switch from nitrogen to air (4). The use of 841 capillaries of 15 cm length and $d_{c}=0.2 \mathrm{~mm}$ inner diameter ensures an homogeneous injection of bubbles of almost equal sizes.

Experiments are performed at ambient temperature and pressure $\left(T=20^{\circ} \mathrm{C}\right.$ and $P=P_{\text {atm }}$ ). The liquid used for all experiments is tap water filtered to remove particles larger than $15 \mu \mathrm{~m}$ (5). As a consequence, in the regime considered, gas-liquid interfaces can be considered to be clean (Ellingsen \& Risso 2001). This point has been carefully validated by measuring the terminal velocity for single bubbles. The main physical properties of the system are summarized in table 1.

### 2.2. Measurements of gas volume fraction and bubble velocity

The gas volume fraction $\alpha$ and the average vertical bubble velocity $\left\langle V_{z}\right\rangle$ are measured by means of a dual-tip optical fibre probe (RBI Instrumentation) which is introduced at the centre of the column (7). A threshold just higher than the noise level is first applied on the raw signal to define the binarized signal. An example of raw and
(a)

(b)


Figure 1. (Colour online) (a) Experimental installation and (b) imaging set-up.
binarized signals obtained for each fibre is presented in figure 2 . Then, the volume fraction is determined from

$$
\begin{equation*}
\alpha=\frac{\sum \Delta t_{y i}}{t_{a q c}} \tag{2.1}
\end{equation*}
$$

| $\rho_{L}$ | 998.2 | $\mathrm{~kg} \mathrm{~m}^{-3}$ |
| :---: | :---: | :---: |
| $\mu_{L}$ | $1.0038 \times 10^{-3}$ | Pa s |
| $\rho_{G}$ | 1.2 | $\mathrm{~kg} \mathrm{~m}^{-3}$ |
| $\mu_{G}$ | $18 \times 10^{-6}$ | Pa s |
| $\sigma$ | $73 \times 10^{-3}$ | $\mathrm{~N} \mathrm{~m}^{-1}$ |
| $D_{L}$ | $2.1 \times 10^{-9}$ | $\mathrm{~m}^{2} \mathrm{~s}^{-1}$ |
| $H e$ | $4.05 \times 10^{9}$ | Pa |
| $P^{\text {sat }}$ | 2337 | Pa |
| $M_{\mathrm{H}_{2} \mathrm{O}}$ | $18.015 \times 10^{-3}$ | $\mathrm{~kg} \mathrm{~mol}^{-1}$ |
| $M_{\mathrm{O}_{2}}$ | $32 \times 10^{-3}$ | $\mathrm{~kg} \mathrm{~mol}^{-1}$ |
| $x_{\mathrm{O}_{2}}^{G 0}$ | $20.9 \%$ | - |

Table 1. System properties at $T=20^{\circ} \mathrm{C}$ and $P=101325 \mathrm{~Pa}$.


Figure 2. Signals from the optical probe. Symbols: raw signal from first ( + ) and second ( $\times$ ) fibre. Line: binarized signals ( - ).
where $t_{a c q}$ is the acquisition duration, $\Delta t_{y i}$ the residence time of bubble $i$ on the first fibre (see figure 2) and $\Sigma \Delta t_{y i}$ the total time during which the gas phase is detected. The signal acquisition is performed with a sampling frequency of 10 kHz . A good statistical convergence and an overall accuracy better than $2 \%$ is obtained for a recording time larger than 800 s .

The vertical velocity $V_{z i}$ of bubble $i$ is obtained by

$$
\begin{equation*}
V_{z i}=\frac{d_{s}}{\Delta t_{12 i}} \tag{2.2}
\end{equation*}
$$

where $\Delta t_{12 i}$ is the time elapsed between the detection of the bubble interface by the first and the second fibre (as reported in figure 2 ) and $d_{s}$ is the distance between the two fibre tips. The main difficulty of this technique is to match two successive rising fronts corresponding to the piercing of the same bubble. Spurious unrealistic low or large velocity measurements are detected in some cases, especially when two bubbles interact close to the probe. According to the sensitivity study of Riboux (2007), values smaller than $V_{\min }=0.03 \mathrm{~m} \mathrm{~s}^{-1}$ or larger than $V_{\max }=0.7 \mathrm{~m} \mathrm{~s}^{-1}$ have been removed.

### 2.3. Measurement of bubble geometrical characteristics

The most reliable technique to determine the bubble shape is probably to process images obtained by means of a high-speed camera. A classic way to image the bubbles is to use a fixed focal lens with a thin depth of field, as done by Colombet et al. (2011). However, the larger the gas volume fraction, the more numerous are blurred out-of-focus bubbles in the field of view. The use of a fixed focal lens is thus limited to moderate gas volume fractions ( $\alpha \leqslant 15 \%$ ).

The study of collective effects in a dense bubble swarm therefore requires the development and the use of another optical technique. In the present work, we use a telecentric lens, which has the particularity to have a depth of field larger than the column width ( 15 cm ) and a constant magnification factor all along the direction of view. The main advantage is to image bubbles with sharp contours, even in a very dense bubbly flow. The main drawback is that the increase of the field of view results in a significant reduction of the spatial resolution. In addition, it has been possible to follow individual bubbles only on a short distance. For those two reasons, the measurement of the bubble velocity is less accurate and image processing has been specifically used to measure the bubble geometrical characteristics.

The imaging set-up consists of a high-speed CMOS camera (Photron APX, figure $1 b$ ) equipped with a telecentric lens (TC-4M-172 Opto Engineering) to visualize a window of $94 \mathrm{~mm} \times 94 \mathrm{~mm}$ located at the centre of the column at a distance of 150 mm above the injectors tips. The spacial resolution is 5.8 pixel $\mathrm{mm}^{-1}$. The camera is operated at 500 images per second with an exposure time varying from $1 / 20000$ to $1 / 500 \mathrm{~s}$ depending on the lighting intensity. Lighting is supplied by an halogen spot of 1000 W .

The recorded images are processed by using Matlab ${ }^{\circledR}$. The bubble edges are detected by applying a threshold to the raw images in grey levels. The interior of the bubbles is then filled and small aberrant objects detected in the picture are removed. A test of convexity is performed to identify cases for which the detected object corresponds to two superimposed bubbles. It consists of comparing the surface area $S_{o b j}$ of the detected object to the area $S_{\text {conv }}$ of the smallest convex polygon that can contain the object. Only the objects with $S_{o b j} / S_{\text {conv }} \geqslant 0.95$ are retained, the others being discarded. Examples of detected contours are drawn on typical raw images in figure 3 for different gas volume fractions.

The geometrical properties of the bubbles are determined by assuming that the bubbles are oblate spheroids with a minor semi-axis $a$ and a major semi-axis $b$, which are measured from the two-dimensional measured contours. The bubble aspect ratio is defined as $\chi=b / a$. The bubble volume is estimated from $V_{b}=4 \pi b^{2} a / 3$ and its equivalent diameter from

$$
\begin{equation*}
d=\left(8 b^{2} a\right)^{1 / 3} \tag{2.3}
\end{equation*}
$$

The bubble area $S_{b}$ is estimated by (Beyer 1987)

$$
\begin{equation*}
S_{b}=\pi \frac{d^{2}}{4}\left(2 \chi^{2 / 3}+\frac{\chi^{-4 / 3}}{\sqrt{1-\chi^{-2}}} \ln \left(\frac{1+\sqrt{1-\chi^{-2}}}{1-\sqrt{1-\chi^{-2}}}\right)\right) . \tag{2.4}
\end{equation*}
$$

In addition, an indirect determination of the bubble equivalent diameter can be obtained from the dual-tip optical probe by assuming that all of the bubbles have the same size. As recalled by Colombet (2012), for a monodispersed population of bubbles that impact the probe with null angle of attack, $d$ can be expressed as a function of the average chord length $\langle y\rangle$,

$$
\begin{equation*}
d=\frac{3}{2}\langle y\rangle \chi^{2 / 3}, \tag{2.5}
\end{equation*}
$$



Figure 3. (Colour online) Typical images of the bubble swarm with detected bubble contours marked with yellow/light lines: (a) $\alpha=12.2 \%$; (b) $\alpha=23.9 \%$; (c) $\alpha=30.6 \%$; (d) $\alpha=33.9 \%$.
where $\langle y\rangle$ is obtained from optical probe measurements as

$$
\begin{equation*}
\langle y\rangle=\frac{\sum_{1}^{n}\left(V_{z i} \Delta t_{y i}\right)}{n}, \tag{2.6}
\end{equation*}
$$

and $\chi$ from image processing. (Note that the size distribution of the bubbles will be discussed in § 3.1 from the results of image processing.)

### 2.4. Measurement of interfacial area

For a bubble column of total volume $V_{\text {tot }}$, the volumetric interfacial area, $a_{I}=$ $\sum S_{b} / V_{\text {tot }}$, is related to the gas volume fraction, $\alpha=\sum V_{b} / V_{t o t}$, by the relation

$$
\begin{equation*}
a_{I}=\alpha \frac{\sum S_{b}}{\sum V_{b}} . \tag{2.7}
\end{equation*}
$$

| $\alpha(\%)$ | $H(\mathrm{~cm})$ | Lower probe $(\mathrm{cm})$ | Upper probe $(\mathrm{cm})$ |
| :---: | :---: | :---: | :---: |
| $\alpha<11$ | 76 | 14.0 | 69.5 |
| $11 \leqslant \alpha<21$ | 64.2 | 14.0 | 55.0 |
| $21 \leqslant \alpha<31$ | 35.2 | 5.8 | 34.8 |
| $31 \leqslant \alpha<32$ | 29.5 | - | - |
| $\alpha \geqslant 32$ | 19.4 | - | - |

Table 2. Liquid height $H$ at $\alpha=0$ and elevation of the oxygen probes above capillaries tips.

As indicated above, for each bubble detected, $a$ and $b$ are obtained from the images used to determine the bubble volume $V_{b}$ and surface $S_{b}$. The volume fraction $\alpha$ is given by the optical probe. Then, the interfacial area $a_{I}$ is determined by using (2.7).

### 2.5. Measurement of mass transfer

The concentration $C(z, t)$ of oxygen dissolved in water at time $t$ and elevation $z$ is measured by means of fast response probes: Clark-type microsensors (Unisense Ox50). The technique is based on the measurement of the intensity of the electric current between an anode and an oxygen-reducing cathode, which is proportional to the oxygen concentration. Calibration of oxygen probes is performed for each experiment. Since the probe response is linear on the whole range of concentration considered, a calibration is performed in situ by using the signal measured at the beginning (anoxic water) and the end (saturated water) of each experiment. The relative uncertainty on oxygen concentration measurements is $\pm 2 \%$. In the present configuration, as shown in figure 1(6), two oxygen probes have been placed at two different elevations $z$, which are reported in table 2.

As shown in Colombet et al. (2011), due to the moderate height of the bubble column $(\leqslant 70 \mathrm{~cm})$, the oxygen saturation concentration in the water is almost unaffected by the variation of hydrostatic pressure ( $6.4 \%$ ) or by the depletion of the oxygen concentration within the bubbles during the mass transfer ( $6 \%$ ). Moreover, the dilution of oxygen in the bubbles induced by liquid-to-gas transfer of nitrogen at the beginning of the experiments can also be neglected ( $1.3 \%$ ). Consequently, the oxygen mass saturation concentration $C^{*}$ can be considered as constant along the $z$ axis and equal to its value at the upper surface where the pressure is equal to that of the atmosphere $\left(P=P_{\text {atm }}\right)$, so that

$$
\begin{equation*}
C^{*}=x_{\mathrm{O}_{2}}^{G 0} \rho_{\mathrm{H}_{2} \mathrm{O}} \frac{M_{\mathrm{O}_{2}}}{M_{\mathrm{H}_{2} \mathrm{O}}} \frac{\left(P-P^{s a t}\right)}{H e} \approx 9.08 \mathrm{mg} \mathrm{l}^{-1} \tag{2.8}
\end{equation*}
$$

with $x_{\mathrm{O}_{2}}^{G 0}$ the molar fraction of oxygen in the gas phase (dry air), $\rho_{\mathrm{H}_{2} \mathrm{O}}=\rho_{L}$ the density of water $\left(\mathrm{kg} \mathrm{m}^{-3}\right), M$ the molar masses $\left(\mathrm{kg} \mathrm{mol}^{-1}\right), P^{\text {sat }}$ the vapour pressure of water in the bubbles ( Pa ) and He the Henry constant for oxygen in water $(\mathrm{Pa})$. Equation (2.8) results from Henry's law for oxygen in water and Raoult's law for water in air with activity and fugacity coefficients equal to unity for both equilibria, assuming that the liquid is essentially composed of water.

The classical 'gassing-out' method is used to determine the time scale of the transfer of oxygen from the bubbles to the water. This method consists in first bubbling nitrogen gas in the column in order to remove the oxygen that is initially naturally present in water. Next, without changing the inlet gas flow rate in order
to not disturb the dynamics of the bubble swarm, air is suddenly injected instead of nitrogen. The concentration of dissolved oxygen $C$ then increases until it reaches the saturation concentration $C^{*}$.

The moderate size of the column and the bubble-induced turbulence both contribute to an efficient liquid mixing so that the liquid phase can be assumed to be perfectly mixed for each horizontal slice of the bubble column. Moreover, owing to the large gas volume fractions and interfacial areas considered in this work, the vertical mass flux of dissolved oxygen generated by the axial mixing can be neglected compared with the oxygen flux coming from the bubbles. In such conditions, the variation of the concentration of dissolved oxygen along the bubble column is given by

$$
\begin{equation*}
\frac{\partial C(z, t)}{\partial t}=\frac{k_{L} a_{I}}{(1-\alpha)}\left(C^{*}-C(z, t)\right), \tag{2.9}
\end{equation*}
$$

where $k_{L}$ is the liquid-side mass transfer coefficient and $a_{I}$ the interfacial area. In the present configuration, the only reason for which $C$ depends on $z$ comes from the delay corresponding for the time taken by the bubble to reach a given elevation $z$. In the following the time origin is shifted by $z /\left\langle V_{z}\right\rangle$ so that the concentration no more depends on $z$ and the signals provided by the two oxygen probes are synchronized.

The analysis of the measured concentrations requires to account for the response time $\tau_{p}$ of the probes (Letzel et al. 1999; Martin, Montes \& Galan 2007). For this purpose, the oxygen probe is assumed to behave as a first-order system

$$
\begin{equation*}
\frac{\partial C_{p}}{\partial t}=\left(1 / \tau_{p}\right)\left(C-C_{p}\right), \tag{2.10}
\end{equation*}
$$

where $C$ is the real concentration and $C_{p}$ is the value provided by the probe. The response time of each probe has been measured and found close to $\tau_{p}=0.8 \mathrm{~s}$. Solving (2.10) and (2.9) for a sudden increase of the mass concentration from 0 to $C^{*}$ at $t=0$, it yields

$$
\begin{equation*}
\frac{C_{p}}{C^{*}}=1-\frac{1}{\left(\tau-\tau_{p}\right)}\left(\tau \mathrm{e}^{-t / \tau}-\tau_{p} \mathrm{e}^{-t / \tau_{p}}\right), \tag{2.11}
\end{equation*}
$$

where the time scale $\tau$ is related to the mass transfer coefficient $k_{L}$ by

$$
\begin{equation*}
\tau=\frac{(1-\alpha)}{k_{L} a_{I}} \tag{2.12}
\end{equation*}
$$

### 2.6. Homogeneity of the bubble swarm

Our purpose is to study a stable bubble column in which there is no gradient of volume fraction and no large-scale liquid motions induced by buoyancy. The use of an array of capillary tubes guarantees that the bubbles are uniformly injected at the bottom of the column. However, increasing the gas volume fraction may lead to the development of an instability and to the transition to a churn flow. The onset of the instability depends on both the liquid height $H$ in the column and the gas volume fraction. For $H=70 \mathrm{~cm}$, the flow is stable up to approximately $\alpha=10 \%$. For larger values of $\alpha$, the liquid height has been reduced in order to keep a stable flow. The chosen values of $H$ are reported in table 2. With this choice, the free surface at the top of the column remains still and the gas volume fraction turns out to be uniform all over the column. Figure 4 compares the superficial gas velocity $J_{G}=\alpha \times\left\langle V_{z}\right\rangle$ obtained from $\left\langle V_{z}\right\rangle$ and $\alpha$ measured by the optical probe and the


FIGURE 4. Superficial gas velocity from gas volume fraction and bubble average rising velocity measured by the dual-tip optical probe $J_{G}=\alpha \times\left\langle V_{z}\right\rangle$ versus superficial gas velocity from measured gas flow rate $J_{G}=Q_{G} / S$.
superficial gas velocity $J_{G}=Q_{G} / S$ obtained from a gas flow rate $Q_{G}$ measured from the flowmeters. The good agreement obtained between these two estimations for all gas volume fractions investigated $(0.45 \leqslant \alpha \leqslant 33.9 \%)$ confirms the homogeneity of the gas distribution over the column.

Another departure to the flow homogeneity may come from the fact that the bubbles need a certain distance to reach their terminal velocity and that mass transfer needs a certain time to attain a steady state. Considering a clean spherical bubble starting from rest, the relaxation time scale of the bubble velocity can be estimated by $\tau_{V} \approx d^{2} /\left(72 v_{L}\right) \approx 0.06 \mathrm{~s}$, which corresponds to a distance $3 \tau_{V} V_{z} \approx 5.4 \mathrm{~cm}$. Concerning the mass transfer, Figueroa \& Legendre (2010) found a transient time $\tau_{C} \approx 10\left(d^{3} \chi / 8\right)^{1 / 3} / V_{z}$ for $R e=300, S c=10$ and $\chi=1.2$. In our case, this leads to $\tau_{C} \approx 0.04 \mathrm{~s}$ and $\tau_{C} V_{z} \approx 1.3 \mathrm{~cm}$. It is therefore reasonable to consider that the flow and the mass transfer are fully developed at the location of the first oxygen probes, which is at least 5.8 cm above the capillaries.

## 3. Characterization of the bubble dynamics

In this section, the bubble dynamics is characterized in terms of bubble size, velocity, deformation, interfacial area and relevant dimensionless numbers. The results obtained by means of a telecentric lens are systematically presented together with those of Colombet et al. (2011), who used a fixed focal lens in the same experimental set-up for $0.45 \leqslant \alpha \leqslant 16.5 \%$. In figures $5-7,9(b)$ and $10(a)$, the errorbars indicate the uncertainty related to the image resolution on the measurement of bubble size and to the measurement of $\alpha$. In figures $9(a), 10(b)$ and 11 , errorbars indicate the uncertainty related to the measurement of $\langle d\rangle$ by considering an uncertainty of $\pm 0.02 \mathrm{~m} \mathrm{~s}^{-1}$ on the determination of the average bubble velocity $\left\langle V_{z}\right\rangle$.

### 3.1. Equivalent diameter and interfacial area

Figure 5 shows the evolution of the average bubble equivalent diameter $\langle d\rangle$ measured from image processing (2.3) as a function of $\alpha(\mathrm{O}, \bullet)$. The standard deviation of


Figure 5. Average bubble equivalent diameter as a function of the gas volume fraction: - image processing with a telecentric lens; $O$, image processing with a fixed focal lens by Colombet et al. (2011); ×, dual-tip optical probe measurements from average bubble chords (2.5); - - -, (3.1); -_, (3.2); -•-•-, dynamic bubble formation model of Gaddis \& Vogelpohl (1986). Inset: $\log -\log$ representation of $\left(\langle d\rangle-d_{0}\right) / d_{0}$ versus $\alpha$.


Figure 6. Interfacial area (2.7) versus gas volume fraction: ©, this work; O, from Colombet et al. (2011); - - -, (3.3); - , (3.4).
the equivalent diameter measured by image processing is found to range between 11 and $21 \%$ of the average value. The bubbles are therefore almost monodisperse and (2.5) can also be used to estimate the bubble diameter from optical probe signals. The values determined by this method are also plotted in figure $5(\times)$. Despite the strong assumptions made, including that the probe is considered to be ideal (Kiambi et al. 2001; Vejrazka et al. 2010) and that all of the bubbles impact the probe with a null angle of attack, the difference between the two experimental techniques is less than $14 \%$.


Figure 7. Average bubble velocity against the gas volume fraction. Dual-tip optical probe measurements from this work (©), Colombet et al. (2011) (O), Riboux et al. (2010) (*). PTV by image processing from Colombet et al. (2011) (ם); -_, (3.5).

The bubble diameter is observed to increase with $\alpha$ because of the process of bubble formation and detachment from the capillaries. At a very low gas volume fraction, the bubble formation can be considered as quasi-static and the bubble size is controlled by the equilibrium between buoyancy and capillary forces at the tip of the capillaries. The diameter is then given by the Tate law, $d_{T}=\left[6 \sigma d_{c} /(\Delta \rho g)\right]^{1 / 3}=2.07 \mathrm{~mm}$, as confirmed by the measurement of the detachment of a single bubble by Riboux et al. (2010). When increasing the inlet gas velocity $u_{c}$, the balance of the forces acting on a bubble involves drag and added-mass forces (Gaddis \& Vogelpohl 1986; Duhar \& Colin 2006). For the entire range of gas volume fraction considered here, the Weber number based on the capillary inner diameter, $W e_{c}=\rho_{L} u_{c}^{2} d_{c} / \sigma$, stays much lower than two so that the jet regime is never reached and the bubble generation corresponds either to the static regime of formation or to the dynamic one (Mersmann 1977). Knowing the gas flow rate through each capillary, the bubble diameter can be estimated by using the model of Gaddis \& Vogelpohl (1986). The predictions of this model, which are reported in figure 5, show the same trend as the experimental results but with an underestimation of approximately $20 \%$. This discrepancy can be due to a collective effect of the bubbles on the formation process and to bubble coalescence that may take place just above the capillary tip as observed by Manasseh, Riboux \& Risso (2008).

A $\log -\log$ representation (see the inset in figure 5) reveals that the evolution of $\langle d\rangle$ is well described by the succession of two power laws:

$$
\begin{gather*}
\frac{\langle d\rangle-d_{0}}{d_{0}} \approx 15 \alpha \quad \text { for } \alpha \leqslant 2.3 \%  \tag{3.1}\\
\frac{\langle d\rangle-d_{0}}{d_{0}} \approx 2.3 \alpha^{0.5} \quad \text { for } \alpha>2.3 \% \tag{3.2}
\end{gather*}
$$

where $d_{0}=2.1 \mathrm{~mm}$ is the value for a single bubble detaching in the static regime from one capillary $(\alpha=0)$.

Figure 6 shows the evolution of the interfacial area as a function of the gas volume fraction. It is found to regularly increase with $\alpha$ according to the following empirical power laws

$$
\begin{array}{ll}
\frac{a_{I}}{a_{I 0}} \approx 0.402 \alpha^{0.85} & \text { for } \alpha \leqslant 2.3 \% \\
\frac{a_{I}}{a_{I 0}} \approx 0.336 \alpha^{0.8} & \text { for } \alpha>2.3 \% \tag{3.4}
\end{array}
$$

where $a_{I 0}=S_{b 0} / V_{b 0}=3011 \mathrm{~m}^{-1}$ is the surface-to-volume ratio of a single bubble detaching in the static regime.

### 3.2. Bubble velocity

### 3.2.1. Average velocity

During the last decade, many works have investigated the velocity of bubbles rising in a swarm (Rusche \& Issa 2000; Zenit, Koch \& Sangani 2001; Garnier et al. 2002; Riboux et al. 2010). All of these studies report a significant decrease of the average bubble vertical velocity as the gas volume fraction increases.

Figure 7 shows the average vertical bubble velocity $\left\langle V_{z}\right\rangle$ as a function of $\alpha$. The present results obtained with a dual optical probe ( $\bullet$ ) are compared with those of Riboux et al. (2010) (*) and Colombet et al. (2011) (O) that were obtained with the same technique, and to those of Colombet et al. (2011) ( $\square$ ) that were determined by image processing with a fixed focal lens. The velocity obtained from image processing is slightly lower, probably because the detected bubbles are not far enough from the column wall. However, all of the results obtained with an optical probe collapse onto a master curve of equation

$$
\begin{equation*}
\left\langle V_{z}\right\rangle=V_{z 0}[0.28+0.72 \exp (-15 \alpha)]^{0.5} \tag{3.5}
\end{equation*}
$$

where $V_{z 0}=0.32 \mathrm{~m} \mathrm{~s}^{-1}$ is the rise velocity of an isolated bubble formed on a single capillary in the quasi-static bubbling regime, measured by Riboux et al. (2010). It is remarkable that a single simple correlation is able to describe the evolution of the average bubble velocity on a such large range of gas volume fraction ( $0.45 \leqslant$ $\alpha \leqslant 29.6 \%$ ) along which $\left\langle V_{z}\right\rangle$ is reduced by almost a factor of two (from 0.32 to $0.17 \mathrm{~m} \mathrm{~s}^{-1}$ ).

It is important to stress that all empirical relations relating the properties of the gas phase to the gas volume fraction that have been introduced above ((3.1)-(3.5), (4.1), (4.3) and (4.5)) may depend on the particular system of gas injection used here and are therefore not universal. However, they will be of great interest to analyse and discuss the results of the present work in the following.

### 3.2.2. Velocity fluctuations

Figure 8 shows the variances of the bubble velocity fluctuations. Let us first discuss the variance of the velocity signal provided by the dual optical probe from this work (O) for $\alpha$ up to $30 \%$ and from Riboux et al. (2010) (*) for $\alpha$ up to $12 \%$. As noted by Riboux et al. (2010), if the dual optical probe is an accurate technique to measure the average vertical bubble velocity, it does not provide a reliable value of the variance of the bubble vertical velocity. The reason comes from the fact that the considered bubbles are oblate spheroid which move with oscillating velocity and orientation.


Figure 8. Variances of bubble velocity against gas volume fraction. Variance obtained from the dual optical probe in this work (O) and by Riboux et al. (2010) (*). Variances of the vertical velocity ( $\square$ ) and the horizontal velocity (■) measured in this work from particle tracking on images taken with a fixed focal lens.

The fluctuations that are recorded by the dual probe are thus a complex combination of the fluctuations of the vertical velocity, orientation and shape. For that reason, the measured variance is observed to depend on the exact probe geometry. The values obtained by Riboux et al. (2010) with a distance between the two fibre tips of 0.5 mm is indeed significantly larger than that obtained in the present work with a fibre tip separation of 1 mm . However, the variance provided by the dual optical probe can be used to characterize the evolution with the gas volume fraction of the overall energy of agitation of the bubbles in the vertical direction. It was already noticed that the bubble vertical agitation keeps a constant value up to a gas volume fraction of around $10 \%$ by Martínez-Mercado, Palacios-Morales \& Zenit (2007) and Riboux et al. (2010), which suggested that the energy of bubble agitation remains controlled by wake instabilities. The present results seem to show that this result holds up to $\alpha=30 \%$.

In order to have a more complete description of the bubble agitation, we have also determined the velocity variance of the horizontal and the vertical bubble velocity fluctuations by PTV based on images taken with a fixed focal lens. As stated before, this imaging technique already used by Colombet et al. (2011) is limited to moderate volume fractions. The corresponding results are also plotted in figure 8 for $\alpha$ up to $16 \%$. Both the horizontal and the vertical variances are found to be almost constant, $\left\langle v_{z}^{\prime 2}\right\rangle \approx 0.003 \mathrm{~m}^{2} \mathrm{~s}^{-2}$ and $\left\langle v_{x}^{\prime 2}\right\rangle \approx 0.0075 \mathrm{~m}^{2} \mathrm{~s}^{-2}$, for $\alpha$ up to $10 \%$. As shown by Ellingsen \& Risso (2001), the horizontal component of the fluctuant velocity of an isolated bubble evolves as $v_{x}^{\prime}=\omega A_{x} \cos (\omega t)$. For the present bubble size, Riboux (2007) measured an angular frequency of $\omega=29 \mathrm{rad} \mathrm{s}^{-1}$ and a path amplitude $A_{x}$ varying from 3.5 to 4.9 mm , which gives a variance $\left\langle v_{x}^{\prime 2}\right\rangle=\left(A_{x} \omega\right)^{2} / 2$ from 0.005 to $0.01 \mathrm{~m}^{2} \mathrm{~s}^{-2}$, in agreement with the values found here at moderate volume fraction. When $\alpha$ is increased beyond $10 \%$, the vertical variance remains constant, while the horizontal one decreases down to match the vertical value around $\alpha=12 \%$. Such a decrease of the horizontal fluctuation of the dispersed phase has already been reported in a solid/liquid fluidized bed by Aguilar Corona (2008) and Aguilar Corona,


Figure 9. (a) Bubble Reynolds number ( $\operatorname{Re}=\left\langle V_{z}\right\rangle\langle d\rangle / v_{L}$ ) and (b) Eötvös number (Eo= $\Delta \rho g\langle d\rangle^{2} / \sigma$ ) versus gas volume fraction: $O$, method using a fixed focal lens; $\bullet$, method using a telecentric lens; $\mathbf{\Delta}$, result for a single bubble from Riboux et al. (2010); __, Reynolds number determined form fitted data ((3.1), (3.2) and (3.5)); - - -, Eötvös number determined form fitted data ((3.1) and (3.2)).

Zenit \& Masbernat (2011). It may result from hindrance effects on bubble paths when increasing $\alpha$.

### 3.3. Bubble Reynolds, Eötvös and Weber numbers

In order to fully characterize the present flow regime, it is useful to consider the values taken by the relevant dimensionless numbers in the range of volume fractions investigated. These values can be computed either from the raw values of the measured dimensional quantities or from the empirical fits proposed in the previous sections. In the following figures, plots systematically represent raw data whereas lines corresponds to values obtained from fitted data.

Figure 9(a) shows the Reynolds number, $\operatorname{Re}=\left\langle V_{z}\right\rangle\langle d\rangle / v_{L}$. It first increases from 670 to 780 as $\alpha$ increases from 0 to $2.5 \%$ and then keeps a constant value as $\alpha$ is further increased. The constance of the Reynolds number for $\alpha \geqslant 2.5 \%$ results from the fact


Figure 10. (a) Bubble average aspect ratio and (b) Weber number ( $W e=\rho_{L}\left\langle V_{z}\right\rangle^{2}\langle d\rangle / \sigma$ ) versus gas volume fraction: $O$, method using a fixed focal lens; $\bullet$, method using a telecentric lens; $\mathbf{\Delta}$, result for a single bubble from Riboux et al. (2010); __, aspect ratio estimated from (3.6) (Legendre, Zenit \& Velez-Cordero 2012); - - -, Weber number determined form fitted data ((3.1), (3.2) and (3.5)).
that the increase in the bubble diameter (figure 5) is compensated by the decrease of the rise velocity (figure 7). A similar result was observed for a volume fraction up to $10 \%$ by Martínez-Mercado et al. (2007), who also used a bank of capillaries to inject the bubbles. This is an interesting property of this type of experimental set-up, which allows the volume fraction to be varied while keeping the Reynolds number constant.

Figure $9(b)$ shows the Eötvös number: Eo $=\Delta \rho g\langle d\rangle^{2} / \sigma$. As expected from the evolution of $\langle d\rangle$, it regularly increases from 0.5 to 3.2 as $\alpha$ varies from 0 to $30 \%$.

Figure $10(a)$ presents the mean bubble aspect ratio, $\langle\chi\rangle$, which is found to slightly decrease from 1.7 to 1.4. The bubble deformation is known to be controlled by both the Weber number (Moore 1965) and the Morton number (Legendre et al. 2012). Here, since we are considering a single system of fluids with constant physical properties, the Morton number is constant: $M o=g \nu_{L}^{4} \rho_{L}^{2} \Delta \rho / \sigma^{3}=2.5 \times 10^{-11}$. The measured Weber


Figure 11. Experimental drag coefficient (3.7) against gas volume fraction: ©, method using a fixed focal lens; - method using a telecentric lens. Drag coefficient for a single bubble of same equivalent diameter rising at its terminal velocity (3.8): $\square$, Mendelson (1967); $\nabla$, Comolet (1979); $\Delta$, Dijkhuizen et al. (2010). Drag coefficient accounting for the collective effect of the bubbles: -_, Wallis (1961) (3.9); - - -, Ishii \& Chawla (1979) (3.10); . ....., Garnier et al. (2002) (3.11); - - -, Roghair et al. (2011) (3.12).
number, $W e=\rho_{L}\left\langle V_{z}\right\rangle^{2}\langle d\rangle / \sigma$, is plotted in figure $10(b)$. It is found to decrease from approximately 3.25 down to 1.8 . Since the Reynolds number is almost constant, the Weber number turns out to be proportional to $\left\langle V_{z}\right\rangle$. The decrease of the average aspect ratio, by approximately $30 \%$, is of the same order as that of We, and both $\langle\chi\rangle$ and We keep an almost constant value for $\alpha>15 \%$. These results are in good agreement, within $14 \%$, with the relation proposed by Legendre et al. (2012) for a single bubble at low Morton number:

$$
\begin{equation*}
\chi=\frac{1}{1-\frac{9}{64} W e} . \tag{3.6}
\end{equation*}
$$

### 3.4. Collective effect on bubble drag coefficient

We consider now the evolution of the bubble drag coefficient $C_{d}$ with the gas volume fraction in order to analyse the collective effect of the bubbles on their rise velocity. Here $C_{d}$ is determined from the balance between drag and buoyancy forces as

$$
\begin{equation*}
C_{d}=\frac{4}{3} \frac{\Delta \rho}{\rho_{L}} \frac{g\langle d\rangle}{\left\langle V_{z}\right\rangle^{2}}, \tag{3.7}
\end{equation*}
$$

where $g$ is the acceleration of gravity, the average equivalent diameter $\langle d\rangle$ is measured from image processing and the average rise velocity $\left\langle V_{z}\right\rangle$ from the dual-tip optical probe. The experimental results are shown in figure $11(\mathrm{O}, \bullet)$ as a function of the gas volume fraction. Here $C_{d}$ is observed to increase from 0.26 for $\alpha=0.45 \%$ and $\langle d\rangle \approx 2.5 \mathrm{~mm}$ to 2.4 at $\alpha=34 \%$ and $\langle d\rangle \approx 5 \mathrm{~mm}$.

In order to disentangle the effect of the bubble size to that of the gas volume fraction, it is interesting to compare the present results with those obtained for single rising bubbles of the same diameter. The drag coefficient $C_{d_{0}}$ of a deformed single
bubble at terminal velocity is commonly estimated by (Tomiyama et al. 1998)

$$
\begin{equation*}
C_{d_{0}}=\frac{8}{3} \frac{E o}{c_{1}+c_{2} E o} \tag{3.8}
\end{equation*}
$$

with $c_{1}=4$ and $c_{2}=1$ according to Mendelson (1967), $c_{1}=4.28$ and $c_{2}=1.02$ according to Comolet (1979) and $c_{1}=19 / 3$ and $c_{2}=2 / 3$ for $R e>600$ with air/water systems according to Dijkhuizen et al. (2010).

The corresponding values are represented by empty squares and triangles in figure 11. Starting from similar values at low gas volume fractions, $C_{d}$ and $C_{d_{0}}$ quickly diverge as $\alpha$ increases. In the present experiments, the increase of $C_{d}$ turns out to mainly result from hydrodynamic bubble interactions. The collective effect of bubbles is really important and leads to a drag coefficient 2.4 times larger than that of an isolated bubble at $\alpha=34 \%$.

In the literature devoted to bubbly flows, numerous relations have been proposed to describe the evolution of the drag coefficient with the gas volume fraction. Considering air bubbles injected through a porous sparger in a column of 9.5 cm diameter filled with a soapy water solution, Wallis (1961) proposed the following correlation, for $3<\alpha<30 \%$

$$
\begin{equation*}
C_{d}=C_{d_{0}}(1-\alpha)^{-2}, \tag{3.9}
\end{equation*}
$$

which was later used for one-dimensional gas-liquid modelling (Wallis 1969, p. 52). We have computed $C_{d}$ using relation (3.9) with $C_{d_{0}}$ from Mendelson (1967). The corresponding values are represented by a plain line in figure 11. They are in fairly good agreement with the present measurements.

Applying a mixture viscosity model to their experimental results, Ishii \& Chawla (1979) (see also Ishii \& Zuber 1979) found the following correction to account for the effect of the gas volume fraction on the bubble drag coefficient:

$$
\begin{equation*}
C_{d}=\frac{C_{d_{0}}}{(1-\alpha)}\left(\frac{1+17.67[f(\alpha)]^{6 / 7}}{18.67 f(\alpha)}\right)^{2} \quad \text { with } f(\alpha)=(1-\alpha)^{1.5} \tag{3.10}
\end{equation*}
$$

This relation is also reported in figure 11 (dashed-dotted line) by using the expression proposed by Mendelson (1967) for $C_{d_{0}}$. It predicts an evolution of $C_{d}$ that is close to that of Wallis (1961) so that it is difficult to conclude which is in best agreement with the present results.

Garnier et al. (2002) experimentally investigated a homogeneous air/water bubbly flow in the presence of a co-current liquid flow for volume fractions up to $30 \%$ and Reynolds numbers from 300 to 500 . They results led to

$$
\begin{equation*}
C_{d}=C_{d_{0}}\left(1-\alpha^{1 / 3}\right)^{-2} \tag{3.11}
\end{equation*}
$$

Using again the expression proposed by Mendelson (1967) for $C_{d_{0}}$, this relation (dotted line in figure 11) is found to considerably over-predict the effect of the gas volume fraction upon the drag coefficient compared with the present results.

Roghair et al. (2011) performed numerical simulations of a bubble swarm in a periodic cubic domain for $1 \leqslant E o \leqslant 5,4 \times 10^{-12} \leqslant M o \leqslant 2 \times 10^{-9}$ and $\alpha \leqslant 45 \%$. From their results, they proposed the following relation

$$
\begin{equation*}
C_{d}=C_{d_{0}}\left(1+\frac{18}{E o} \alpha\right) \tag{3.12}
\end{equation*}
$$

where $C_{d_{0}}$ is given by the relation proposed by Dijkhuizen et al. (2010). This relation suggests that the collective effect of the bubbles on the drag coefficient may depend on other parameters than the gas volume fraction, such as the Eötvös number. The corresponding values of $C_{d}$ are represented by a dashed line in figure 11. They are approximately $30 \%$ higher than the present experimental data, just at the limit of the measurement uncertainty.

## 4. Mass transfer

In this section, the measured mass transfer coefficients and Sherwood numbers are first presented as a function of the gas volume fraction. Then, they are compared with available correlations for a single bubble rising in a liquid at rest. Finally, they are compared with transfer rates expected in a highly turbulent flow. In figures 14 and 15 , errorbars indicate the uncertainty related to the measurement of the interfacial area $a_{I}$, the gas volume fraction $\alpha$, the bubble equivalent diameter $\langle d\rangle$ by considering an uncertainty of $\pm 10 \% \tau$ on the determination of the mass transfer time scale $\tau$.

### 4.1. Experimental results

The time evolutions of the oxygen concentration are presented in figure 12 for $\alpha=1.46 \%(a), \alpha=15.1 \%(b)$ and $\alpha=26.9 \%(c)$. In this figure, the time origin has not been shifted by $z /\left\langle V_{z}\right\rangle$ so that the signal of the upper probe is delayed compared with the first. The least-squares method is used to fit each set of experimental data by (2.11) in order to obtain the transfer time scale $\tau$. The corresponding fitting curves, represented by lines in figure 12, describe accurately the experimental results, confirming that the assumptions made about the probe response and the fact that the flow is well mixed are fulfilled.

A total of 38 experimental runs have been conducted in the range of $0.7 \leqslant \alpha \leqslant$ $29.6 \%$. The corresponding values of $\tau$ are reported in figure 13 together with the 29 values measured by Colombet et al. (2011) in the range $0.45 \leqslant \alpha \leqslant 16.5 \%$. The time necessary to reach the saturation is significantly affected by the void fraction since it decreases by more than one order of magnitude between $\alpha=1 \%$ and $\alpha=30 \%$. Such a strong decrease is expected from the strong increase of the interfacial area with $\alpha$ (figure 6). As it is clearly visible in the $\log -\log$ plot proposed in the inset of figure 13 , the experimental values of $\tau$ nicely follow a simple power law,

$$
\begin{equation*}
\tau \approx \tau_{0} \alpha^{-0.8} \text { with } \tau_{0}=2.22 \mathrm{~s} \tag{4.1}
\end{equation*}
$$

In order to analyse the collective effect of the bubbles on the mass transfer, we have to consider the mass transfer coefficient by unit of area, $k_{L}$. The experimental value of $k_{L}$ is obtained from the measured values of $\tau, \alpha$ and $a_{I}$ by

$$
\begin{equation*}
k_{L}=\frac{(1-\alpha)}{\tau a_{I}} \tag{4.2}
\end{equation*}
$$

Combining relations (3.4) for $a_{I}$ and (4.1) for $\tau$, the following simple empirical relation is found for the mass transfer coefficient, for $\alpha>2.3 \%$

$$
\begin{equation*}
k_{L} \approx k_{L_{0}}(1-\alpha) \quad \text { with } k_{L_{0}}=4.45 \times 10^{-4} \mathrm{~m} \mathrm{~s}^{-1} \tag{4.3}
\end{equation*}
$$

Figure 14(a) shows the evolution of the experimental values of $k_{L}$ as a function of the gas volume fraction. The decrease is considerably lower compared with that of $\tau$, which indicates that most of the evolution of the total rate of transfer results from the trivial effect of the augmentation of the interfacial area and justifies the efforts made to obtain an accurate determination of $a_{I}$.


Figure 12. (Colour online) Typical measured time evolutions of the concentration of dissolved oxygen of (a) $\alpha=1.46 \%$, (b) $\alpha=15.1 \%$ and (c) $\alpha=26.9 \%$ : ©, lower probe; O upper probe; - - - and -


Figure 13. Time scale of the mass transfer versus the gas volume fraction: $\square$, lower probe; $\Delta$, upper probe; O, Colombet et al. (2011) (using a single oxygen probe); ——, experimental fit (4.1). Inset: log-log representation.

To go further in the analysis of the physical mechanism underlying the mass transfer, we have to make dimensionless the rate of mass transfer by introducing the Sherwood number

$$
\begin{equation*}
S h=\frac{k_{L}\langle d\rangle}{D_{L}} \tag{4.4}
\end{equation*}
$$

where $D_{L}$ is the diffusion coefficient of dissolved oxygen in water. Figure $14(b)$ shows the evolution of the experimental Sherwood number as a function of $\alpha$, which using empirical fits (3.1), (3.2), (4.3) and (4.4) can be described by the following empirical relation, for $\alpha>2.3 \%$

$$
\begin{equation*}
S h \approx S h_{0}(1-\alpha)\left(1+2.3 \alpha^{0.5}\right) \quad \text { with } S h_{0}=445 \tag{4.5}
\end{equation*}
$$

The increase of $\langle d\rangle$ almost compensates for the decrease of $k_{L}$ so that $S h$ turns out to increase moderately with the gas volume fraction, its values (4.5) remaining between 600 and 750 in the whole range of $\alpha$ investigated.

The present results therefore suggest that the collective effect of the bubbles has a relatively weak influence on the interfacial mass transfer considering the huge effect observed on the transfer time scale $\tau$. However, it is difficult to conclude from the sole evolution of the Sherwood number since variations of bubble size, velocity and shape are associated with variation of gas volume fractions. In next section, we will compare the present results with those corresponding to a single bubble in the same flow regime and with the same geometrical properties.

### 4.2. Comparison with a single bubble rising in a liquid at rest

In most studies of mass transfer in bubble columns, the rate of transfer is estimated by using Higbie's penetration theory (Higbie 1935),

$$
\begin{equation*}
k_{L}=\frac{2}{\sqrt{\pi}} \sqrt{\frac{D_{L}}{t_{c}}} \tag{4.6}
\end{equation*}
$$



Figure 14. Mass transfer coefficient (a) and Sherwood number (b) versus gas volume fraction. Experiments: $\square$, this work (average values); O, Colombet et al. (2011); $\quad$, empirical fits ((4.3) and (4.5)). Predictions for an isolated bubble: $\boldsymbol{\Delta}$, single bubble ((4.12) and (4.13) with the parameters measured for an isolated bubble detached from a single capillary); — • - (4.7) (Boussinesq 1905); ......., (4.8) (Winnikow 1967); - - -, (4.9) (Takemura \& Yabe 1998); _-, (4.10) (Colombet et al. 2013); -_, (4.12) and (4.13) (Figueroa \& Legendre 2010).
where $t_{c}$ is taken equal to $\langle d\rangle /\left\langle V_{z}\right\rangle$. In fact this is equivalent to the analytical solution obtained by Boussinesq (1905) by considering that the flow around the bubble can be approximated by the potential flow and a very thin concentration layer on the bubble

$$
\begin{equation*}
S h=\frac{2}{\sqrt{\pi}} P e^{1 / 2}, \tag{4.7}
\end{equation*}
$$

where $P e=\langle d\rangle\left\langle V_{z}\right\rangle / D_{L}$ is the Péclet number. This solution is thus valid in the limit of large $R e$ and $P e$.

Various improvements have been proposed to account for the effect of bubble deformation or finite Reynolds number upon the mass transfer from a single bubble.


Figure 15. Comparison of the measured Sherwood number with mass transfer models for turbulent flows. Present experiments: same legend as in figure 14; (4.14) and (4.15) with - $c_{1}=2 / \sqrt{\pi}$ (Kawase et al. 1987), $---c_{1}=0.523$ (Linek et al. 2004), - • $c_{1}=0.4$ (Lamont \& Scott 1970).

Considering the flow approximation of Moore (1963), Winnikow (1967) derived the following expression that includes the effect of the Reynolds number:

$$
\begin{equation*}
S h=\frac{2}{\sqrt{\pi}}\left[1-\frac{2.89}{\sqrt{R e}}\right]^{1 / 2} P e^{1 / 2} \tag{4.8}
\end{equation*}
$$

Measuring the mass transfer of almost spherical millimetre-sized bubbles from volume variations, Takemura \& Yabe (1998) proposed the following relation,

$$
\begin{equation*}
S h=\frac{2}{\sqrt{\pi}}\left[1-\frac{2}{3} \frac{1}{\left(1+0.09 R e^{2 / 3}\right)^{3 / 4}}\right]^{1 / 2}\left(2.5+P e^{1 / 2}\right) \tag{4.9}
\end{equation*}
$$

which was found to be in good agreement with both experiments and numerical simulations at moderate $R e$ numbers and large $P e$.

Recently, considering numerical results from various previous works, Colombet et al. (2013) proposed the following relation that is valid for a spherical bubble whatever the value of Re and Pe ,

$$
\begin{equation*}
S h=1+\left[1+\left(\frac{4}{3 \pi}\right)^{2 / 3}\left(2 P e_{\max }\right)^{2 / 3}\right]^{3 / 4} \tag{4.10}
\end{equation*}
$$

where $P e_{\max }$ is the Péclet number based on the maximal velocity $U_{\max }$ of the liquid at the interface instead of the bubble velocity $V_{z}$, which is obtained from the correlation proposed by Legendre (2007),

$$
\begin{equation*}
\frac{U_{\max }}{V_{z}}=\frac{1}{2} \frac{16+3.315 R e^{1 / 2}+3 R e}{16+3.315 R e^{1 / 2}+R e} \tag{4.11}
\end{equation*}
$$

When $P e$ tends to zero, relation (4.10) tends to the diffusive solution in the absence of flow ( $S h=2$ ) while it tends towards the Boussinesq solution when $R e$ and $P e$ become very large.

The effect of the bubble deformation has been studied by Lochiel \& Calderbank (1964), who considered the potential flow around a spheroidal bubble. They proposed to correct the Boussinesq expression by the introduction of a function $f$ of the aspect ratio $\chi$,

$$
\begin{equation*}
\operatorname{Sh}(\chi)=\frac{2}{\sqrt{\pi}} P e^{1 / 2} f(\chi) \tag{4.12}
\end{equation*}
$$

The validity of this solution has been recently discussed by Figueroa \& Legendre (2010), who proposed the following expression

$$
\begin{equation*}
f(\chi)=0.524+0.88 \chi-0.49 \chi^{2}+0.086 \chi^{3} \tag{4.13}
\end{equation*}
$$

which is based on the results of direct numerical simulations, and proved to be valid for $500<(\chi / 8)^{1 / 3} R e<1000, \nu_{L} / D_{L}>100$ and $1 \leqslant \chi \leqslant 3$.

The values of $k_{L}$ predicted by all of these expressions derived for an isolated bubble are reported in figure $14(a)$ while the corresponding values of $S h$ are reported in figure $14(b)$. Because the Reynolds number remains almost constant and the bubble shape does not evolve so much in the present experiments, the predictions of all of these correlations are close to each other. Moreover, these predictions are all in agreement with the experiments, considering the measurement uncertainty.

We can therefore conclude that the mass transfer in a homogeneous bubble swarm at high Péclet number is almost independent of the gas volume fraction. It has been proved to remain similar to that of a single bubble rising in a fluid at rest up to a volume fraction of $30 \%$. This conclusion is in agreement with the trends of the numerical simulations of Roghair (2012).

### 4.3. Comparison with the interfacial transfer in highly turbulent flows

The bubbles generate strong velocity fluctuations in the liquid phase. It is thus interesting to compare the rate of transfer measured here with that induced at a plane interface by a turbulence of similar intensity. It has been shown that turbulent eddies can enhance the mass transfer by causing the renewal of the liquid close to the interface (Magnaudet \& Calmet 2006). Considering that the timescale $t_{c}$ of renewal of the liquid at the interface is proportional to $\left(\nu_{L} /\left\langle\epsilon_{L}\right\rangle\right)^{1 / 2}$, where $\epsilon_{L}$ is the rate of dissipation of the turbulence, (4.6) gives

$$
\begin{equation*}
S h=c_{1}\left(\frac{d\left\langle\epsilon_{L}\right\rangle^{1 / 4}}{v_{L}^{3 / 4}}\right) S c^{1 / 2}, \tag{4.14}
\end{equation*}
$$

where $S c=v_{L} / D_{L}$ is the Schmidt number. Several values have been proposed for the prefactor $c_{1}: 0.4$ (Lamont \& Scott 1970), 0.523 (Linek et al. 2004) or $2 / \sqrt{\pi}$ (Kawase et al. 1987).

In an homogeneous bubbly flow, Riboux et al. (2010) showed that the rate of dissipation of the energy is given by

$$
\begin{equation*}
\left\langle\epsilon_{L}\right\rangle \approx \frac{\Delta \rho}{\rho_{L}} \alpha\left\langle V_{z}\right\rangle g . \tag{4.15}
\end{equation*}
$$

According to (4.15), $\left\langle\epsilon_{L}\right\rangle$ ranges from 0.02 to $0.5 \mathrm{~m}^{2} \mathrm{~s}^{-3}$ for the range of gas volume fraction considered here. The Sherwood numbers given by relation (4.14) are
plotted in figure 15. They are clearly not in agreement with the present measurements. Combining (4.15) and (4.14), it yields

$$
\begin{equation*}
S h=c_{1}\left(E o^{3} / M o\right)^{1 / 8} \alpha^{1 / 4} R e^{1 / 4} S c^{1 / 2} \tag{4.16}
\end{equation*}
$$

The evolution of Sherwood number with the gas volume fraction predicted by this relation $\left(\alpha^{1 / 4}\right)$ is not compatible with the experimental result. Moreover, the predicted evolution with the Reynolds number $\left(R e^{1 / 4}\right)$ is contradictory to the scaling expected considering the evolution for an isolated bubble ( $R e^{1 / 2}$ ).

This analysis confirms that the mass transfer in the bubble column is controlled by the mass transfer around a single bubble in fluid at rest. The fact that the liquid agitation may play a negligible role in the mass transfer at a bubble interface has already been noticed by Alves et al. (2006), who investigated the case of a single bubble in a turbulent flow with a dissipation rate of one order of magnitude smaller than in the present configuration.

## 5. Discussion

Hydrodynamic interactions between bubbles have a strong effect on the average bubble rise velocity, which is found to decrease strongly when increasing the gas volume fraction. The analysis of the interactions between two rising bubbles in a liquid at rest reveals opposite effects depending on the relative position of the bubbles. For two bubbles rising in line, the drag force on the trailing bubble is diminished (Yuan \& Prosperetti 1994; Harper 1997; Ruzicka 2000; Hallez \& Legendre 2011), so that vertical interactions between bubbles should increase the average bubbles velocity in a bubble swarm. On the other hand, the drag coefficient of two bubbles rising side by side is increased (Legendre, Magnaudet \& Mougin 2003; Hallez \& Legendre 2011), so that transversal interactions between bubbles should decrease the average bubble velocity. In a two-dimensional high-Reynolds-number swarm of bubbles confined between two vertical plates (Bouche et al. 2012), vertical interactions are predominant and both the average and the variance of the vertical bubble velocity is observed to increase with the gas volume fraction. The main difference between this configuration and the present one is that turbulence cannot develop because of the wall friction. In a three-dimensional unconfined bubble swarm, hydrodynamic interactions between bubble wakes cause a strong attenuation of individual wakes (Risso et al. 2008). The combination of the wake attenuation with the existence of an intense agitation of both the bubbles and the liquid phase reduces considerably the vertical entrainment by bubbles and explains why the hindering effect is predominant when the gas volume fraction increases. More surprising is the weak influence of hydrodynamic interactions on the variance of the vertical bubble agitation, which is observed to remain close to that of an isolated bubble. Even if bubble path oscillations become erratic, the fluctuant energy of their motion seems still controlled by wake instabilities.

The major finding of the present work is the absence of any significant collective effect of the bubbles on the mass transfer up to volume fraction of $30 \%$. This result is not valid for any dispersed two-phase flow. Collective effect on the mass transfer are known to exist for a long time (Ranz \& Marshall 1952). In the 1960s and the 1970s, many experimental works on mass (or heat) transfer in fixed or fluidized bed measured an increase of the Sherwood (or Nusselt) number with the particle volume fraction $\alpha_{S}$ (Ranz \& Marshall 1952; Rowe \& Claxton 1965; Littman \& Silva 1970; Turner \&

Otten 1973; Gunn \& Souza 1974; Miyauchi, Kataoka \& Kikuchi 1976; Gunn 1978). More recently, numerical simulations have confirmed this trend: Massol (2004) for $0 \leqslant$ $R e \leqslant 300,0.72 \leqslant v / D \leqslant 2$ and $\alpha_{S} \leqslant 60 \%$; and Deen et al. (2012) for $36 \leqslant \operatorname{Re} \leqslant 144$, $v / D=0.8$ and $\alpha_{S}=30 \%$, who found results in agreement with Gunn (1978). The point is therefore to understand the absence of collective effect in homogenous bubbly flows.

First, let us discuss the mechanism of mass transfer for a single rising bubble. For large Reynolds and Péclet numbers, Figueroa \& Legendre (2010) showed that the mass transfer mainly takes place across a thin diffusive layer located at the front part of the bubble, where the flow is potential. The thickness $\delta_{D}$ of the concentration boundary layer can be estimated from (Boussinesq 1905)

$$
\begin{equation*}
\frac{\delta_{D}}{d} \approx \frac{\sqrt{\pi}}{2} P e^{-1 / 2} \tag{5.1}
\end{equation*}
$$

In the present experiments, the Péclet number is around $3.5 \times 10^{5}$ and $\delta_{D}$ is of the order of $10^{-3} d(\approx 5 \mu \mathrm{~m})$. In order that the solution for a single bubble can apply, two conditions must be fulfilled. First, the average flow in the close surrounding of each bubble have to be similar to that of a isolated bubble. Second, liquid velocity fluctuations must not penetrate within the concentration boundary layer. Experimental investigations of the flow around a bubble immersed within an homogeneous bubble swarm (Risso \& Ellingsen 2002; Roig \& Larue de Tounemine 2007; Risso et al. 2008) have shown that the first condition is satisfied; in particular, at high bubble Reynolds number, the flow in front of the bubble is well described by the potential solution for a single bubble. The second condition requires that both the distance $\delta_{x}$ between the interfaces of neighbour bubbles and the size of the smallest turbulent eddies $\eta_{K}$ are large compare to the thickness $\delta_{D}$ of the concentration boundary layer.

If bubbles were arranged on a periodic face-centred cubic network, the minimum distance between two bubble interfaces is given by

$$
\begin{equation*}
\frac{\delta_{x}}{d}=\left(\frac{1}{\sqrt{2}}\left(\frac{2 \pi}{3 \alpha}\right)^{1 / 3}-1\right) \tag{5.2}
\end{equation*}
$$

which gives $\delta_{x} \approx 0.35 d \approx 1.6 \mathrm{~mm}$ for $\alpha=30 \%$.
As suggested by Riboux et al. (2010), the Kolmogorov microscale of the bubbleinduced turbulence, which corresponds to the size of the smallest turbulent eddies, can be roughly estimated by

$$
\begin{equation*}
\eta_{K}=\left(\frac{v_{L}^{3}}{\left\langle\epsilon_{L}\right\rangle}\right)^{1 / 4} \tag{5.3}
\end{equation*}
$$

where the average dissipation rate $\left\langle\epsilon_{L}\right\rangle$ is determined from (4.15) and gives $\eta_{K} \approx$ $10^{-2} d \approx 50 \mu \mathrm{~m}$ for $\alpha=30 \%$. Both $\delta_{x}$ and $\eta_{K}$ are therefore much larger than $\delta_{D}$ and the second condition is satisfied.

With a Péclet number of 1070, the results of Roghair (2012) did not show any collective effect. However, in the cases considered by Massol (2004) ( $P e=600$ ) and Deen et al. (2012) ( $P e=115$ ), the mass transfer was observed to depend on the volume fraction of the dispersed phase. This confirms that a large enough Péclet number is necessary so that the mass transfer is not influenced by hydrodynamic interactions.

## 6. Conclusions

Thanks to an original method of imaging using a telecentric lens and a dual optical probe, the properties of the gas phase have been measured in an homogenous swarm of bubble up to a volume fraction of $30 \%$. In particular, the bubble deformation is found to be in good agreement with the correlation proposed by Legendre et al. (2012) for a single bubble. Also, the average bubble velocity is observed to strongly decrease with $\alpha$ and found to be in agreement with the correlations of Wallis (1961) and Ishii \& Chawla (1979). Even if some open questions remain concerning the physical mechanism responsible for the increase of the drag coefficient, available correlations are reliable to predict the deformation and the average bubble rise velocity in an homogenous bubble swarm at large Reynolds number. The bubble fluctuating velocity has also been characterized. Surprisingly, no significant influence of the gas volume fraction on the variance measured by means of the dual optical probe was observed. Hydrodynamic interactions between bubbles make the bubble path oscillations irregular, but they do not seem to modify the overall amount of fluctuating energy, which remains controlled by the instability of the wake behind each bubble.

The total mass transfer of oxygen from the bubbles to the liquid has been measured by means of the gassing-out method. Thanks to the determination of the total interfacial area by imaging, the mass transfer rate by unit of area and the Sherwood number have been obtained. Remarkably, the Sherwood number is found to be very close to that of a single bubble rising at the same velocity. The reason lies in the fact that the mass flux occurs in a very thin layer located in front of the bubble. Owing to the large value of the Péclet number $\left(>10^{5}\right)$, the distance between the interfaces of the bubbles and the smallest turbulent eddies are much larger than the thickness of the concentration boundary layer. Consequently, the flow within this layer is not affected by the presence of the other bubbles. Moreover, the mixing generated by the bubble-induced agitation of the liquid ensures that the dissolved oxygen is homogeneous everywhere out of this layer. For the mass transfer the conditions are therefore equivalent to those of a single bubble rising in a fluid at rest and at uniform concentration. Correlations for the Sherwood number established for single rising bubbles are therefore relevant to predict the mass transfer in a homogenous bubble column up to a volume fraction of $30 \%$, provided that the bubble Reynolds and Péclet numbers are large enough. This conclusion is consistent with the results obtained experimentally for a similar system by Colombet et al. (2011) for gas volume fractions lower than $17 \%$ and with the numerical simulations of Roghair (2012) for a Péclet number around 1000. Results obtained at lower Péclet number in fluidized beds however showed an increase of the Sherwood number compared with that of a single particle. There is probably a lower limit below which mass transfer in a dispersed two-phase flow depends on the volume fraction. The determination of this limit, which probably depends on parameters such as the nature of the dispersed phase or the Reynolds number, requires further investigations.

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