1	TITLE	
2	Design of a Thermoelectric Generator with Fast Transient Response	
3		
4	ABSTRACT	
5	Thermoelectric modules are currently used both in Peltier cooling and in Seebeck mode for electricity	
6	generation. The developments experienced in both cases depend essentially on two factors: the	
7	thermoelectric properties of the materials that form these elements (mainly semiconductors), and the	
8	external structure of the semiconductors. Figure of Merit Z is currently the best way of measuring the	
9	efficiency of semiconductors, as it relates to the intrinsic parameters of the semiconductor: Seebeck	
10	coefficient, thermal resistance, and thermal conductivity. When it comes to evaluating the complete	
11	structure, the Coefficient of Performance (COP) is used, relating the electrical power to the thermal power of	
12	the module. This paper develops a Thermoelectric Generator (TEG) structure which allows minimising the	
13	response time of the thermoelectric device, obtaining short working cycles and, therefore, a higher working	
14	frequency.	
15		
16	KEYWORDS	
17	Thermoelectric Generator; High Dynamic Response; Geometry Design; Thermoelectric Model	
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22			
23	HIGHLIGHTS		
24	- Development of a thermoelectric structure which verifies the model used.		
25	- Design and manufacture of a thermoelectric module with a short response time.		
26	- Relationship between geometry of the thermoelectric system and its performance.		
27	- Experimental testing of time to change of state on the thermoelectric module.		

2 NOMENCLATURE

2	0	Discipated heat flow (M)
3	Q _H	Dissipated heat flow (W)
4	Qc	Absorbed heat flow (W)
5	1	Electrical current (A)
6	Z	Figure of merit (K ⁻¹)
7	R	Electrical resistance (Ω)
8	R _s	Semiconductor's electrical resistance (Ω)
9	R _m	Electrical resistance of the junction metal in the semiconductors (Ω)
10	l	Length of a semiconductor and pellet (m)
11	С	Equivalent electrical capacity (F)
12	T _c	Cold side temperature (°C)
13	T _h	Hot side temperature (°C)
14	T ₀	Room temperature that surrounds the thermal structure (°C)
15	T _{1,2,3,4,5,6}	Temperatures in the different structure interfaces (°C)
16	n	Total number of pellets in the thermoelectric structure
17	P _E	Electrical power (W)
18		
19	GREEK SYMB	OLS:
20	\overline{V}	Three-dimensional partial derivative
21	α	Seebeck coefficient ($\mu V \cdot K^{-1}$)
22	в	Thermal diffusivity (m ² ·s ⁻¹)
23	σ	Electrical conductivity ($\Omega \cdot m^{-1}$)
24	ρ	Electrical resistivity ($\Omega \cdot m$)
25	К	Thermal conductivity (mW·cm ^{-1} ·K ^{-1})
26	κ ₀	Thermal conductivity of material in contact with air (mW·cm ⁻¹ ·K ⁻¹)
27	K _{cc}	Thermal conductivity of the ceramic cold plate (mW·cm ⁻¹ ·K ⁻¹)

1	K _{ch}	Thermal conductivity of the ceramic hot plate (mW·cm ^{-1} ·K ^{-1})
2	K _m	Thermal conductivity of the metal contacts between semiconductors (mW·cm ^{-1} ·K ^{-1})

- κ_s Thermal conductivity of the semiconductor used (mW·cm⁻¹·K⁻¹)

1 **1 INTRODUCTION**

Cooling devices are very important in everyday life. For example, a patient with diabetes should always travel 2 with their insulin to a certain temperature level. Another example is the application of thermoelectricity in 3 cryogenic surgery in oncology. Therefore, thermoelectric devices should be compact, easily portable and 4 flexible. Moreover, it would be very interesting that these devices were long-lasting and maintenance free. 5 6 When the application is not related with cooling or heating new needs are created, such as modifying the 7 geometry of the thermoelectric structure and to develop new thermoelectric materials for manufacturing 8 9 these devices. The creation of these new modules is possible and depends on the design and the development process when thermoelectric modules are being performed. 10 11 These thermoelectric structures are usually classified according to its power generation and its cooling 12 13 temperature range. A thermoelectric structure can perform as a refrigerator of electro-optic components [1], also can be used for turning waste heat into power [2], and it can be applied in several medical areas [3], for 14 example. Due to current market needs, the efficiency of thermoelectric structures is a factor that should be 15 improved. When a thermoelectric device is used as a generator (Seebeck mode), the scope of use is even 16 more restricted [4]. It is highly important to maximize the contribution of each thermo-element; in other 17 words, a high density of thermoelectric generation is needed. 18 19 20 Figure of Merit Z or the dimensionless equivalent ZT is not the only factor that determines the material choice, but it is the most important. Precisely the efficiency of thermoelectric devices is determined by the 21 Figure of Merit, which is defined by the $Z = \alpha^2 \sigma / \kappa$ ratio. It is common to use the term *Power Factor* to 22

23 designate $\alpha^2 \sigma T$ or $\alpha^2 \sigma$, since this term only contains the electronic properties, while κ is always a large 24 contribution of thermal network.

25

One of the main problems in the conversion of solar energy into thermal or electrical power is the low efficiency of converters based on semiconductors. The reason is that a considerable part of energy flows

1 through phonon subsystem (thermal oscillations of crystal lattice), which are not involved in the power

2 generation.

3

For obtaining the *COP* (Coefficient of Performance) of a thermoelectric cell working as cooler we should know the absorbed heat by the thermoelectric structure Q_c and the electrical power output of the system P_{ε} .

6

7

$$COP = \frac{Q_c}{P_F}$$
(1)

8

9 To achieve an increased efficiency, amongst other factors, the geometry of the semiconductor must be taken into account. However, the accurate characterization of a thermoelectric device, either working in Peltier or in 10 Seebeck mode, is rather difficult [5]. When it comes to modelling a semiconductor, the figure of merit is 11 required. There are several methods to obtain this measurement but these should be applied with a degree of 12 13 caution [6]. Generally speaking, modelling is considered from a point of view in which the properties of materials used in the thermoelectric structure have an average value according to the working conditions -14 15 normally the temperature difference between faces-, or also Finite Element Models that lead to the division of the analyzed structure in a number of finite parts, bearing in mind separately that each part has certain 16 17 specific properties.

18

At present, frequency behavior is rarely studied in the modelling of thermoelectric structures. However, the importance of the temporal response in the performance of the thermoelectric structures is shown in [7,8], for example. There are applications that are related with cell miniaturization and the inertia in the transient response that require a frequency study.

23

For all these reasons, in this paper a discrete model based on electric analogy is used to develop and
 manufacture a thermoelectric generator with a configurable time response that is hardly expect reached by
 any commercial thermoelectric module.

- 1 Developed thermoelectric device can be used in an industrial scenario for converting the wasting heat in
- 2 electrical power, but with the advantage that the transient response can be configured according to the final
- 3 application. Regarding electric analogy used in this paper, this model has been already tested in conventional
- 4 structures [9] and its use allows taking into account the time constant of the thermoelectric system.
- 5

6 2 MODEL USED IN THE DESIGN

7 2.1 BASIC RELATIONS

8 In general, the heat flow in a semiconductor could be expressed according to the laws of heat for the three
9 dimensions:

10

11
$$\nabla(\kappa \nabla T) - T(I \nabla \alpha) + I^2 \rho = 0$$
 (2)

13 In one dimension, this is expressed as:

14

15
$$\frac{\partial}{\partial x} \left(\kappa A \frac{\partial T}{\partial x} \right) - IT \frac{\partial \alpha}{\partial x} + I^2 \rho = 0$$
(3)

16

When considering a steady state system in which the change of internal energy is zero, the model can be
determined as a function of the heat absorbed or dissipated in the external faces of the semiconductor [10]:

20
$$\begin{bmatrix} \alpha l - \kappa & \kappa \\ -\kappa & \alpha l + \kappa \end{bmatrix} \cdot \begin{bmatrix} T_h \\ T_c \end{bmatrix} = \begin{bmatrix} Q_h - \frac{1}{2}l^2 R \\ Q_c + \frac{1}{2}l^2 R \end{bmatrix}$$
(4)

21

Although the simple model above does not define other phenomena which take place in the thermoelectric structure – such as the Thomson effect, or the Nerst effect inside a magnetic field –, nor the different phenomena related to the structure set up and used for supporting the semiconductors, it does reveal the high significance of geometry in the evaluation of the model's coefficients [11].

1 When trying to obtain a change of state where there is a high temperature difference, as ΔT , into another 2 state where there is no temperature variation $\Delta T = 0$, a dynamical study of the thermoelectric device must be 3 carried out.

4

5 2.2 ELECTRICAL ANALOGY MODEL

6 The model is considered in one dimension that provides essential information from the thermal structure. The

7 following model is based on temperature increases between interfaces of the different materials that

8 constitute the thermal structure [12]. The thermal parameters are assumed as constant (they actually depend

9 on temperature, geometry, and other factors). Only the thermal conduction phenomenon will be considered,

assuming that the structure is immersed in an isotropic medium [13].

11

12 In the simple structure it is assumed that there are *n* semiconductors and, therefore, the same number of

metallic contacts between the semiconductors [14]. Both ceramic surfaces will support all the structure as

shown in Figure 1.

15

16 The linear equations that define the thermal structure are:

17

18

 $\kappa_0 \left(T_0 - T_1 \right) - \kappa_{cc} \left(T_1 - T_2 \right) + Q_c = 0 \tag{5}$

19
$$\kappa_{cc}(T_1 - T_2) - \frac{n}{2}\kappa_m(T_2 - T_3) = 0$$
 (6)

20
$$n\left(-\alpha IT_{3}-\frac{1}{2}I^{2}\left(R_{s}+\frac{R_{m}}{2}\right)+\kappa_{s}\left(T_{4}-T_{3}\right)+\frac{1}{2}\kappa_{m}\left(T_{2}-T_{3}\right)\right)=0$$
 (7)

21
$$n\left(\alpha IT_{4} + \frac{1}{2}I^{2}\left(R_{s} + \frac{R_{m}}{2}\right) - \kappa_{s}\left(T_{4} - T_{3}\right) - \frac{1}{2}\kappa_{m}\left(T_{5} - T_{4}\right)\right) = 0$$
(8)

22
$$\frac{n}{2}\kappa_{m}(T_{4}-T_{5})-\kappa_{ch}(T_{5}-T_{6})=0$$
 (9)

23
$$\kappa_{ch}(T_5 - T_6) - \kappa_0(T_6 - T_0) - Q_h = 0$$
 (10)

The analogy between thermal and electrical systems allows us to develop our model as an electrical circuit, in 1 such a way that the heat flow is analogous to an electrical current, and the temperature difference is 2 analogous to a voltage difference [14]. Taking into account the thermal-electrical analogy, the linear 3 equations developed result in the circuit shown in Figure 2. 4 5 6 In any case, the heat flow through a semiconductor element is given by the Laplace operator of the heat flow, 7 as shown in the following equation. If is shown in one dimension, for simplification purposes, it would be [15]: 8 $\nabla T = \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial v^2} + \frac{\partial^2 T}{\partial z^2} \longrightarrow \frac{\partial^2 T(x,t)}{\partial x^2} = \frac{1}{\beta} \cdot \frac{\partial T(x,t)}{\partial t}$ (11)9 10 Note that $\beta = \kappa / (\rho \cdot c)$ is the thermal diffusivity. 11 12 13 Based on the previous expression, we can establish an analogy between the time constant of the thermal structure and the time constant of the electrical circuit [16]. In this way geometrical and transient parameters 14 15 are introduced in the design process, showing its relevancy in the thermoelectric module operation. 16 The electrical time constant equivalent to the performance described above is: 17 18 $RC = \frac{4\ell^2}{\pi^2\beta} = \frac{4\ell^2\rho c}{\pi^2\kappa}$ (12) 19 20 If we multiply and divide by the transversal area of the single module, we will obtain the following expression: 21 22 $RC = \frac{4\ell^2 \rho c}{\pi^2 \kappa} = \frac{4\ell\ell A \rho c}{\pi^2 \kappa A} = \frac{4\ell m c}{\pi^2 \kappa A}$ (13) 23 24 We can identify the terms C = mc and $R = \ell / (\kappa A)$ in Equation (14). 25

3 DEVELOPED SAMPLE DEVICE

2	A practical thermoelectric structure is developed, which reaches specific change times from one state in
3	which there is a steady operation with a specific temperature difference, to another state with zero
4	temperature difference and low inertia. Figure 3 shows the design of the thermoelectric device, in which
5	pellets of special size have been used to achieve a specific operation. Commercial thermoelectric structures
6	have cubical pellets. In this work we have developed pellets with a non-standard geometry, with a dimension
7	of 1.5 \times 1.5 \times 6 (width \times height \times depth, in millimeters). This arrangement of the pellets is aimed for
8	generating electric power [17].
9	
10	Pellet geometry should be optimized by considering the interaction of all system elements, that is, between
11	the pellets and the rest of elements of the thermoelectric structure. For this reason is necessary to consider
12	other factors related to pellets geometry that can influence either directly or indirectly on the system
13	efficiency. These factors considered in our product development are as follows:
14	
15	• <i>Volume of the pellet</i> . This value should be reduced.
16	• Contact surfaces between electrical connections. An additional thermal power is generated in the
17	electrical connections due to the Joule effect. These losses may vary depending on the design of the
18	pellet geometry.
19	• Mechanical design. Not all configurations and shapes of the pellets are valid from the mechanical
20	standpoint. To get a robust thermoelectric module that supports mechanical stress is necessary.
21	• Cost and ease of manufacture. Economical cost may be affected by the pellets geometry, although it
22	depends on the size of the manufactured series. A balance between economical cost and design
23	optimization must be taken into account during the product design.
24	
25	Regarding to the effect in the time domain, in order to obtain a reduction in the transient response between
26	the states, we have added metal conductive elements between the thermoelectric faces [18] in the system.

1 pipe is placed on one side of the structure, as shown in Figure 4. On the other hand, the deactivation time

2 tends to get higher, but by introducing a heat sink these losses are balanced.

3

final application.

28

At a structural level, the greater the distance between the hot face and cold face of the thermoelectric 4 module, the lower the thermal conductance. This increases efficiency (semiconductors with longer length 5 6 than a conventional device working as a Peltier device) [19], but if the length of the semiconductor is 7 increased too much, the Joule effect is also increased, causing a decrease in the system's efficiency. The heat pipe must be able to dissipate all the heat that reaches the cold face coming from the heat flow of 8 the heat source [20]. This results in a notably reduced time constant equivalent to RC with the incorporation 9 of the new elements in the system. 10 11 4 RESULTS 12 13 The equations shown in this article allow to know the thermal system response in function of frequency. Consequently, this aspect allows us to control and improve the time response of the thermoelectric system as 14 is shown in following figures. 15 16 Figure 5 shows the temporal response of output voltage, output current and electrical power supplied by the 17 thermoelectric module when a temperature difference of 35°C is applied between hot and cold faces, and a 18 load of 30Ω is connected to the thermoelectric structure. It should be noted that results are optimal and the 19 20 system reaches the steady state in the shortest time possible. When temperature increases the thermoelectric module is able to maintain a constant current and voltage and consequently the power 21 22 delivered to the load. The time spent by the system until steady state is approximately 4 seconds. Disconnection time is slightly higher, although using heat pipe the transient has been improved. 23 24 A comparison between the experimental results and theoretical simulation is shown in Figure 6. Note that the 25 theoretical model is consistent with the experimental response. Therefore, the expressions developed make 26 possible to modify the transient response of the thermoelectric system in the design process depending on its 27

1 **5 CONCLUSIONS**

- 2 In this work, several design expressions have been developed which provide insight into the system's
- 3 transient response under conditions of smaller thermal inertia, depending on the structural characteristics of
- 4 the thermoelectric module. Based on these expressions, it has been possible to reduce the temporal response
- 5 of the thermoelectric module prior to its manufacturing.
- 6
- 7 Moreover, the thermoelectric structure has been designed, developed and manufactured which verifies the
- 8 theoretical expressions presented in this paper. The thermoelectric module achieves shorter times in state
- 9 changes than a conventional structure, making it suitable for use in applications requiring fast response times.
- 10
- 11 Comparisons between simulation results and experimental results show a good correlation. In addition to
- 12 characterising the thermoelectric structure, the reliability of the model used has been verified, since the
- 13 theoretical values obtained are very similar to the experimental results.

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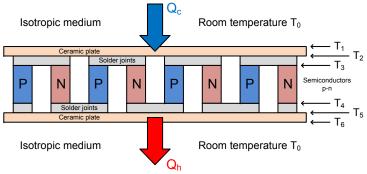
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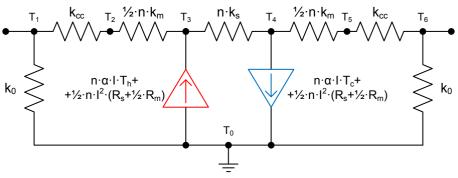
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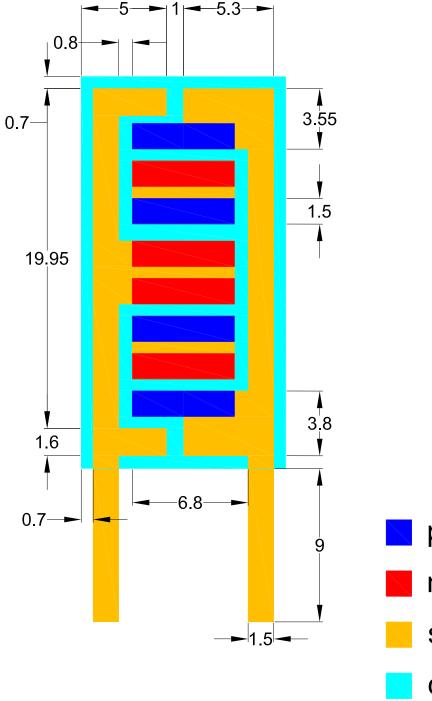
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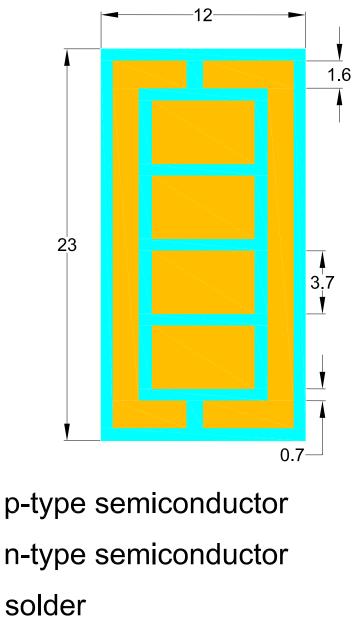
2 FIGURES CAPTION

- 3 Figure 1. Thermoelectric structure diagram with different temperatures considered
- 4 Figure 2. Equivalent electrical circuit
- 5 Figure 3. Thermoelectric structure developed
- 6 Figure 4. Final thermoelectric generator developed with heat pipe and heat sink
- 7 Figure 5. Temporal response of the thermoelectric module
- 8 Figure 6. Comparison between the experimental results ($V_{out exp}$) and theoretical simulation ($V_{out theo}$)









ceramic plate

Figure 4

