

## FLOW CHARACTERISTICS AND STRUCTURES OF THREE-DIMENSIONAL UNSTEADY THERMAL CONVECTION IN A CONTAINER

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### ABSTRACT

In this study, we numerically investigate the flow and thermal characteristics of the three-dimensional thermal convection in a cubic cavity heated below in the gravitational field, concerning about spatially-averaged kinetic energy  $\bar{K}$ , Nusselt number  $Nu$  and flow structure. We assume Prandtl number  $Pr = 7.1$  (water) and Rayleigh number  $Ra = 1.0 \times 10^4 - 3.5 \times 10^5$ . As a result, we have specified two of three important values of the Rayleigh number which demarcate different flow bifurcations and are referred to as the second and third critical Rayleigh numbers  $Ra_{c2}$  and  $Ra_{c3}$ . We have found that  $Ra_{c2}$  and  $Ra_{c3}$  are roughly  $2.6 \times 10^5$  and  $3.1 \times 10^5$ , respectively. We have observed a hysteresis effect upon the value of  $Ra_{c2}$  with chaotic behaviour at  $Ra \approx Ra_{c2}$ , and revealed flow structures. In addition, we investigate the relationship between  $Ra$  and the oscillatory-convection frequency. The increasing rate of the  $\bar{K}_{\text{mean}}$  with increasing  $Ra$  shows a different manner from that of  $Nu_{\text{inflow-ave, mean}}$ . That is, the former is progressive and the latter is asymptotic, as  $Ra$  increases. Both the values of  $\bar{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  in oscillatory flow tend to be smaller than those in steady flow, respectively. Then, there exist small jumps/drops of  $\bar{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  at  $Ra = Ra_{c2}$ .

### INTRODUCTION

Thermal convection has been a key phenomenon for heat and mass transfers. So, a lot of researches have studied about the thermal convection. Also, the thermal convection is treated in such various fields as geology, meteorology, safety aspect of

atomic reactor and manufacturing industries, and becomes important for mass and heat transfers/diffusions and material mixings in many appliances.

In the present study, we consider the three-dimensional thermal convection in one of the most fundamental and simplest containers; that is, a cubic cavity heated below in the gravitational field.

A lot of researchers have studied the cubic cavity. For example, Janssen et al.<sup>(1)</sup> numerically studied bifurcations from steady flow to periodical flow in a cubic cavity. Pallares et al. showed numerically the three-dimensional convection in a cubic cavity at moderate Rayleigh numbers  $Ra$ 's  $< 6.0 \times 10^{4(2)}$ ,<sup>(3)</sup> and showed experimentally and numerically it at high  $Ra$ 's  $< 10^{8(4)}$ ,<sup>(5)</sup>. They reported seven different steady-flow structures; namely, four kinds of single-roll structures (S1, S2, S3 and S7), two kinds of four-roll structures (S5 and S6), and one kind of a toroidal-roll structure (S4)<sup>(3)</sup>. We should note that there are three important values of  $Ra$  which demarcate different flow bifurcations. There are referred to as the first, second and third critical Rayleigh numbers  $Ra_{c1}$ ,  $Ra_{c2}$  and  $Ra_{c3}$ .  $Ra_{c1}$ ,  $Ra_{c2}$  and  $Ra_{c3}$  indicate the transitions to steady flow, oscillatory flow and turbulent flow, respectively. Hirano et al.<sup>(6)</sup> and Brown et al.<sup>(7)</sup> conducted experiments at  $Ra = 10^4 - 10^5$  and  $10^8 - 10^{12}$ , respectively. Crunkleton et al.<sup>(8)</sup> specified the value of  $Ra_{c2}$  for a low Prandtl number  $Pr = 0.008$ , concerning cavities including a cubic one. Recently, Hirata et al.<sup>(9)</sup>, Bennet & Hsueh<sup>(10)</sup> and Valencia et al.<sup>(11)</sup> have numerically shown detailed flow structures at middle and high  $Ra$ 's.

Now, we have the knowledge about thermal convection in the wide range of  $Ra$ . However, the chaotic transition process

from the laminar to turbulent flows has not been clear yet, due to its complexity. So, we numerically investigate the details of the flow and thermal characteristics of the three-dimensional thermal convection in a cubic cavity at the transition range of  $Ra$ . Especially, we consider spatially-averaged kinetic energy  $\bar{K}$ , Nusselt number  $Nu$  and flow structure. Here, we assume incompressible fluid with  $Pr = 7.1$  (water) and  $Ra = 1.0 \times 10^4 - 3.5 \times 10^5$ . Both bottom and top walls are taken to be isothermal, and the bottom temperature is greater than the top temperature. Four vertical walls are conductive. We try to determine the value of  $Ra_{c2}$  and  $Ra_{c3}$ . In addition, we investigate the relationship between the Rayleigh number and the oscillatory frequency of the convective flow at  $Ra > Ra_{c2}$ .

## METHOD

We analyse the convection assuming incompressible fluid with a constant Prandtl number  $Pr = 7.1$  (water) in the gravitational field. Tested Rayleigh number  $Ra$  is fixed to  $1.0 \times 10^4 - 3.5 \times 10^5$ . Both bottom and top walls are taken to be isothermal, and the bottom temperature is greater than the top one. Four vertical sidewalls are conductive. Under these conditions, the flow attains a steady state or oscillatory state with such a flow structure as the S1, S2, S5, S6 or S5<sub>p</sub> after enough time (see later) or attains a chaotic or turbulent state referred to as Chaotic.

## Governing Equation

The dimensionless Navier-Stokes equations with the Boussinesq approximation and an energy equation are as follows;

$$\nabla \cdot \mathbf{u} = 0, \quad (1)$$

$$\frac{D\mathbf{u}}{Dt} = -\nabla p + Pr \Delta \mathbf{u} + Ra Pr T \mathbf{e}_z, \quad (2)$$

and

$$\frac{DT}{Dt} = \Delta T. \quad (3)$$

where  $\mathbf{u}$ ,  $t$ ,  $p$ ,  $T$ ,  $\mathbf{e}_z$  and  $\omega$  denote velocity vector, time, pressure, temperature, the unit vector in the  $z$  direction and (angular) frequency, respectively.

Independent variables are non-dimensionalised as follows;

$$\mathbf{x} = \begin{pmatrix} x \\ y \\ z \end{pmatrix} = \frac{1}{H^*} \begin{pmatrix} x^* \\ y^* \\ z^* \end{pmatrix}, \text{ and } t = \frac{t^*}{H^{*2} / \alpha^*}. \quad (4)$$

As well, dependent variables are non-dimensionalised as follows;

$$\mathbf{u} = \begin{pmatrix} u \\ v \\ w \end{pmatrix} = \frac{H^*}{\alpha^*} \begin{pmatrix} u^* \\ v^* \\ w^* \end{pmatrix}, \quad T = \frac{T^* - T_c^*}{T_h^* - T_c^*}, \text{ and } p = \frac{p^* H^{*2}}{\rho^* \alpha^{*2}}, \quad (5)$$

where  $\alpha^*$ ,  $T_c^*$ ,  $T_h^*$ , and  $\rho^*$  denote the coefficient of thermal diffusivity, a cold top-wall temperature, a hot bottom-

wall temperature and mean density, respectively. A superscript “\*” represents to be dimensional.

## Governing Parameter

Non-dimensional governing parameters are as follows.

$$\text{Rayleigh number: } Ra = \frac{g^* \beta^* (T_h^* - T_c^*)}{\nu^* \alpha^*}. \quad (6)$$

And,

$$\text{Prandtl number: } Pr = \frac{\nu^*}{\alpha^*}. \quad (7)$$

Here,  $\beta^*$ ,  $\nu^*$ , and  $g^*$  denote thermal expansion coefficient, kinetic viscosity, and the gravitational acceleration, respectively.

## Boundary Condition

Boundary conditions for velocity are as follows.

$$u = v = w = 0 \quad (\text{on all walls}) \quad (8)$$

Boundary conditions for pressure are given by the Navier-Stokes equations (2) with the no-slip condition of equation (8).

Boundary conditions for temperature are as follows.

$$T = 1, \quad (\text{at } z = 0) \quad (9)$$

$$T = 0, \quad (\text{at } z = 1) \quad (10)$$

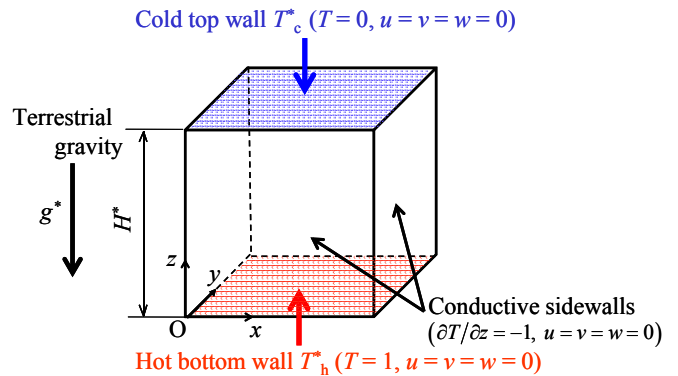
and

$$\frac{\partial T}{\partial z} = -1. \quad (\text{at } x = 0, x = 1, y = 0 \text{ and } y = 1) \quad (11)$$

In equation (11), we consider thermally conductive conditions on four sidewalls.

## Numerical Method

The governing equations (1) — (3) are solved by a finite difference method based on the MAC scheme with a staggered computational grid with a size of  $81^3$ . The present flow is calculated at a time step  $\Delta t = 5.0 \times 10^{-7}$ . Figure 1 shows the computational domain to be analysed, together with the present coordinate system and boundary conditions.



**Fig. 1 Computational domain, together with coordinate system and boundary conditions.**

**Table 1 Accuracy check: a comparison on the Nusselt number averaged over a heated bottom-wall surface between Pallares et al. <sup>(2), (3)</sup> and the present study with an adiabatic-sidewall boundary condition.**

$Ra$	$Pr$	Flow structure	$Nu_{\text{inflow-ave}}$		Error (%)
			Pallares et al.	Present	
$8.0 \times 10^3$	0.71	S2	1.70 <sup>(3)</sup>	1.77	4.12
$8.0 \times 10^3$	1	S1	1.755 <sup>(2)</sup>	1.79	1.99
$6.0 \times 10^4$	0.71	S1	3.13 <sup>(3)</sup>	3.19	1.92

### Global Indicators

As global indicators, we consider such physical quantities as a spatially-averaged kinetic energy  $\bar{K}$  and an inflow-average Nusselt number  $Nu_{\text{inflow-ave}}$ . These are defined as follows.

$$\bar{K} = \frac{1}{V} \iiint_V K dV. \quad (15)$$

Here,  $K$  denotes the kinetic energy defined as

$$K = \frac{1}{2}(u^2 + v^2 + w^2),$$

and  $V$  denotes the volume of the cavity.

$$Nu_{\text{inflow-ave}} = \frac{\iint_{A_{\text{inflow}}} Nu_{\text{local}} dA}{H^2}. \quad (16)$$

Here,  $Nu_{\text{local}}$  denotes a local Nusselt number, and  $A_{\text{inflow}}$  denotes the total area of the bottom-wall surface and the sidewall surfaces where  $Nu_{\text{local}}$  is positive. Because the four sidewalls are conductive, some parts of the sidewalls can allow the inflow of heat as well as the bottom wall. Then,  $Nu_{\text{inflow-ave}}$  represents the total inflow of heat normalised by an area  $H^2$ . We should note that both  $\bar{K}$  and  $Nu_{\text{inflow-ave}}$  are functions of time  $t$ .

## RESULTS AND DISCUSSION

### Numerical Accuracy

In order to confirm the present numerical accuracy, we investigate the flow with an adiabatic-sidewall boundary condition for  $Ra = 8.0 \times 10^3$  and for  $Pr = 0.71$  and 1, in some preliminary computations. These are the same conditions as Pallares et al. <sup>(2), (3)</sup> Table 1 shows a result for the accuracy check: a comparison on the Nusselt number averaged over a heated bottom-wall surface between Pallares et al. <sup>(2), (3)</sup> and the present study. We can confirm a good agreement between them.

### Flow Structures

As shown in Fig. 2, we reveal the flow structures for  $Pr = 7.1$  as a function of  $Ra$ . The results for  $Ra \leq 8.0 \times 10^4$  are by Hirata et al. <sup>(9)</sup> The upper row in the figure corresponds to decreasing  $Ra$ , and the lower row corresponds to increasing  $Ra$ . Specifically speaking, the upper row indicates the flow structure solved using the solution at a higher  $Ra$  as an initial condition, and the lower row is *vi-ce versa*.

In Fig. 2, we can observe four kinds of steady flow structures; namely, two single-roll structures and two four-roll

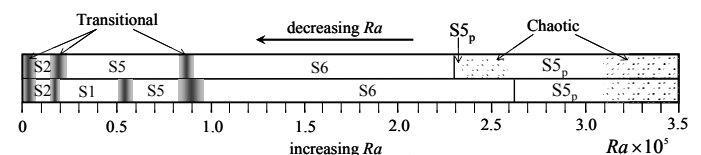
structures (as will be shown latter). These steady flow structures are referred to as S1, S2, S5 and S6, respectively, by Pallares et al. <sup>(3), (4)</sup> In addition to these four steady flow structures, we can observe two kinds of unsteady flow structures, which are referred to as  $S5_p$  and Chaotic. The  $S5_p$  is a periodic flow structure with the S5 at each instance. The Chaotic is not periodic (as will be shown latter).

From the figure, we can specify two of three important values of the  $Ra$  which demarcate different flow bifurcations —  $Ra_{c2}$  and  $Ra_{c3}$ , indicating the transition to oscillatory flow and the transition to chaotic or turbulent flow, respectively. We can find that  $Ra_{c2}$  and  $Ra_{c3}$  are roughly  $2.6 \times 10^5$  and  $3.1 \times 10^5$  respectively. In addition, we can observe a histerisis effect upon the value of  $Ra_{c2}$ , with chaotic behaviour at  $Ra = Ra_{c2}$ . Incidentally,  $Ra_{c1}$  is  $8.0 \times 10^3$ , which indicates the transition to conduction and steady flow.

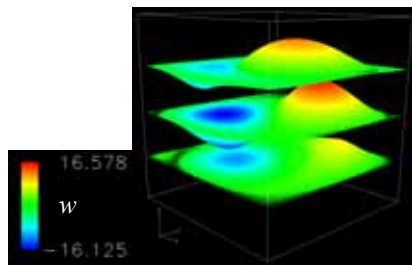
Figures 3, 4, 5 and 6 show the typical flow structures for the S2, S1, S5 and S6, respectively. In each figure, figure (a) represents a perspective view of velocity vectors on the three horizontal plane at  $z = 0.25, 0.5$  and  $0.75$ , where each arrow's colour denotes the value of  $w$  as shown in a colour bar on its left side. As well, figure (b) represents top view of velocity vectors on the horizontal plane at  $z = 0.5$ . Figure (c) represents velocity vectors on a vertical diagonal section, where each arrow's colour denotes the value of  $T$  as shown in a colour bar on its left side. As well, figure (d) represents velocity vectors on the other vertical diagonal section.

The S2 is featured by the single roll with a horizontal axis on a vertical diagonal-section plane. On the other hand, the S1 is featured by the single roll with a horizontal axis on a vertical mid-plane, which are perpendicular to a pair of opposite sidewalls. For both the S2 and the S1, we can clearly recognise a single-roll structure.

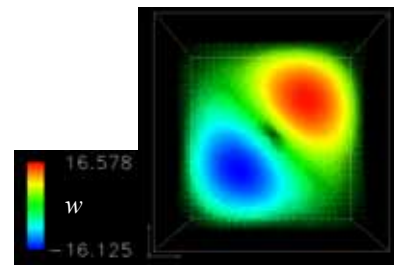
The S5 is featured as both a pair of upward centripetal flows on a vertical diagonal-section plane and a pair of downward centripetal flows on the other vertical diagonal-section plane. On the other hand, the S6 is featured as both a pair of upward centripetal flows on a vertical mid-plane and a pair of downward centripetal flows on the other vertical diagonal-section plane.



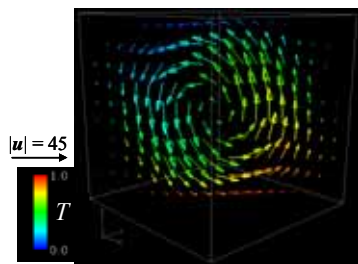
**Fig. 2 Flow structures for  $Ra \leq 3.5 \times 10^5$  and  $Pr = 7.1$ .**



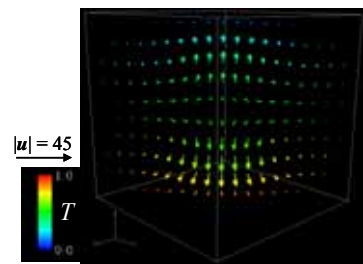
(a) Perspective view of velocity vectors on the three horizontal plane at  $z = 0.25, 0.5$  and  $0.75$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.



(b) Top view of velocity vectors on the horizontal plane at  $z = 0.5$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.

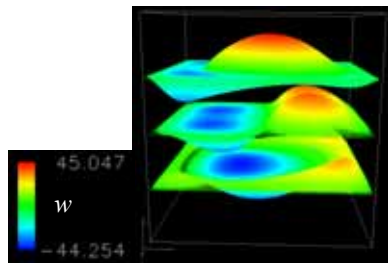


(c) Velocity vectors on a vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

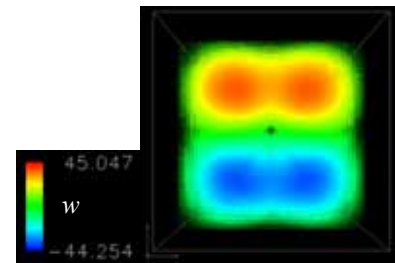


(d) Velocity vectors on the other vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

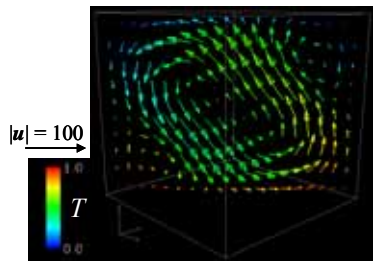
**Fig. 3 Flow structure S2, for  $Ra = 1.0 \times 10^4$  and  $Pr = 7.1$  with increasing  $Ra$ .**



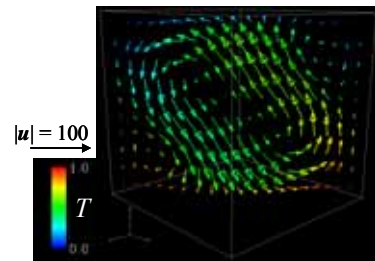
(a) Perspective view of velocity vectors on the three horizontal plane at  $z = 0.25, 0.5$  and  $0.75$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.



(b) Top view of velocity vectors on the horizontal plane at  $z = 0.5$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.

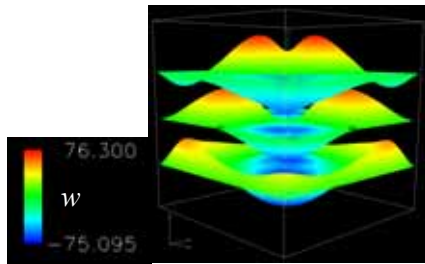


(c) Velocity vectors on a vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

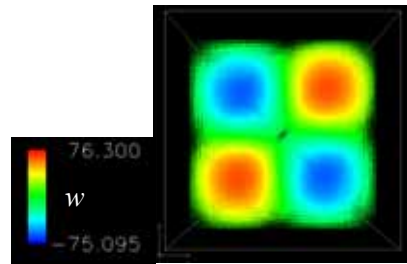


(d) Velocity vectors on the other vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

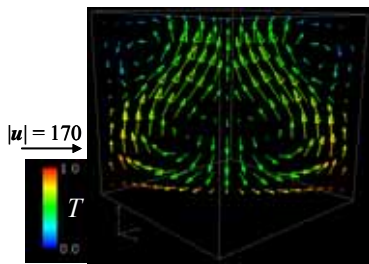
**Fig. 4 Flow structure S1, for  $Ra = 4.0 \times 10^4$  and  $Pr = 7.1$  with increasing  $Ra$ .**



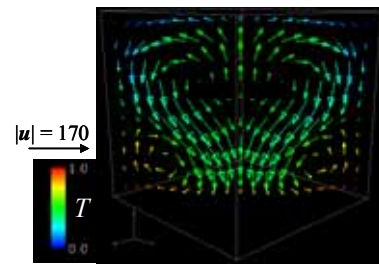
(a) Perspective view of velocity vectors on the three horizontal plane at  $z = 0.25, 0.5$  and  $0.75$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.



(b) Top view of velocity vectors on the horizontal plane at  $z = 0.5$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.

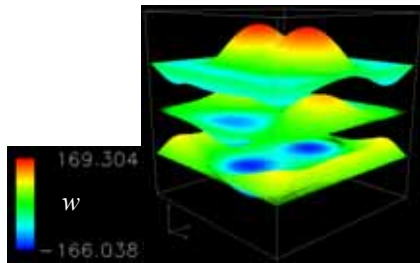


(c) Velocity vectors on a vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

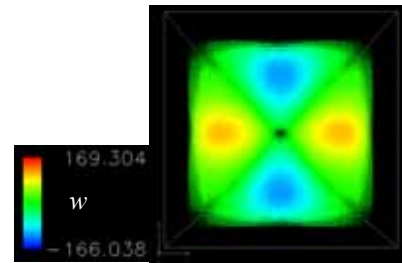


(d) Velocity vectors on the other vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

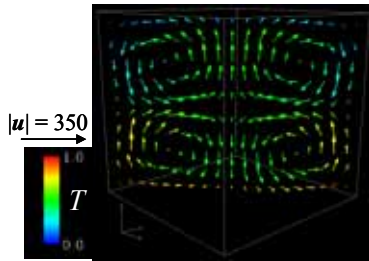
**Fig. 5 Flow structure S5, for  $Ra = 8.0 \times 10^4$  and  $Pr = 7.1$  with increasing  $Ra$ .**



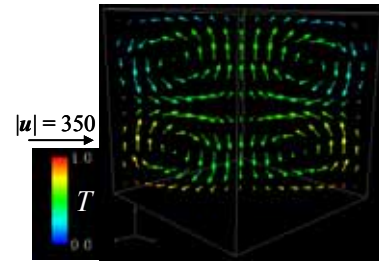
(a) Perspective view of velocity vectors on the three horizontal plane at  $z = 0.25, 0.5$  and  $0.75$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.



(b) Top view of velocity vectors on the horizontal plane at  $z = 0.5$ . Each arrow's colour denotes the value of  $w$  as shown in a colour bar.



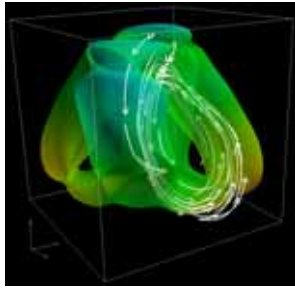
(c) Velocity vectors on a vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.



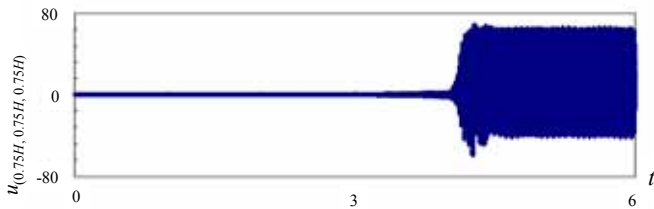
(d) Velocity vectors on the other vertical diagonal section. Each arrow's colour denotes the value of  $T$  as shown in a colour bar.

**Fig. 6 Flow structure S6, for  $Ra = 18.0 \times 10^4$  and  $Pr = 7.1$  with increasing  $Ra$ .**

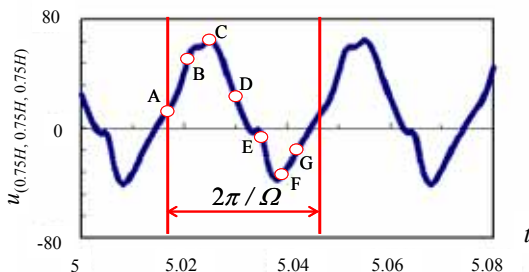
According to Paraller et al.<sup>(4)</sup> both the S5 and S6 are defined as four-roll structures. However, it seems difficult to find out such four-roll structures in Figs. 5 and 6. Figure 7 shows a stream line together with an iso-kinetic surface of  $0.2 \times \bar{K}_{\max}$ , for  $Ra = 8.0 \times 10^4$  and  $Pr = 7.1$ , in order to visualise a typical S5. When we see the iso-kinetic surface, it seems difficult to find out the four-roll structure as well as Fig. 5. The four-roll structure can be clearly visualised by streamlines with low velocity magnitudes near the sidewalls (see Fig.4 in Paraller et al.<sup>(3)</sup>). The streamline in Fig. 7, which has high velocity magnitudes, is considered to represent one of the distorted rolls of the four-roll structure.



**Fig. 7** One of the four rolls in S5 visualised by a stream line for  $Ra = 8.0 \times 10^4$  and  $Pr = 7.1$ , with an iso-kinetic-surface of  $0.2 \times K_{\max}$ .



(a) Overall view (at  $t = 0 - 6.0$ )



(b) Close-up view (at  $t = 5.0 - 5.0894$ )

**Fig. 8** Time history of  $u_{(0.75H, 0.75H, 0.75H)}$ , for  $Ra = 2.65 \times 10^5$  and  $Pr = 7.1$  with increasing  $Ra$  (S5<sub>p</sub>).

### Frequency of S5<sub>p</sub> and Chaotic

Figure 8 shows a time history of  $u_{(0.75H, 0.75H, 0.75H)}$  for  $Ra = 2.65 \times 10^5$  and  $Pr = 7.1$  with increasing  $Ra$ , which corresponds to the S5<sub>p</sub>. Figures (a) and (b) are overall and close-up views, respectively.

We can confirm an exact periodicity with a dominant frequency  $\Omega$ , long time after the beginnings of computations. However, its wave form is far from sinusoidal (see figure (b)).

Supplementary speaking, the points A — G in Fig. 8(b) denote the instants chosen for the flow visualisations in Fig. 9. Now, we consider the flow structure of the S5<sub>p</sub>. Figure 9 shows the consecutive series of flow structures during one period  $2\pi/\Omega$ , for  $Ra = 2.65 \times 10^5$  and  $Pr = 7.1$  with increasing  $Ra$ . Namely, at each instant, we show a perspective view of velocity vectors on the three horizontal planes at  $z = 0.25, 0.5$  and  $0.75$ , where each arrow's colour denotes the value of  $w$  as shown in a colour bar on the right hand. We can confirm that instantaneous flow structure is the same as the S5 at anytime.

Figures 10 and 11 show time histories of  $u_{(0.75H, 0.75H, 0.75H)}$ , and their corresponding spectra, for  $Ra = 2.5 \times 10^5$  and  $Pr = 7.1$  with decreasing  $Ra$ , and for  $Ra = 3.3 \times 10^5$  and  $Pr = 7.1$  with increasing  $Ra$ , respectively. Both the figures represent the Chaotic.

At first, we see Fig. 10. Although figure (a) shows poor periodicity, we can observe a clear spectrum peak in figure (b). Next, we see Fig. 11. Figure (a) shows not an exact periodicity, but a close periodicity. In fact, we can observe a sharp spectrum peak in figure (b). As a result, flow in the Chaotic always fluctuates with one dominant frequency, in spite of the degree of less periodicity.

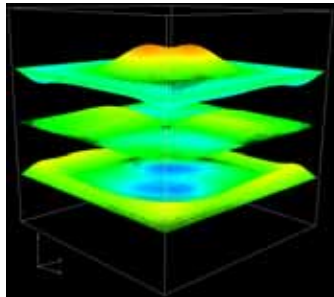
Figure 12 summarises such a dominant frequency. Namely, the figure shows a non-dimensional frequency  $\Omega$  of oscillatory convection in the S5<sub>p</sub> and the Chaotic, plotted against  $Ra$ . We can see that  $\Omega \approx 300$  at  $Ra = 2.28 \times 10^5 - 2.45 \times 10^5$ , and that  $\Omega \approx 200$  at  $Ra = 2.50 \times 10^5 - 3.50 \times 10^5$ . This suggests that the S5<sub>p</sub> at  $Ra = 2.28 \times 10^5 - 2.36 \times 10^5$  is not the same as that at  $Ra = 2.62 \times 10^5 - 3.10 \times 10^5$ , from a viewpoint of oscillatory-convection frequency. Accordingly, the Chaotic at  $Ra = 2.36 \times 10^5 - 2.60 \times 10^5$  shows a transition feature between the two different S5<sub>p</sub>, and is considered to be different from the Chaotic at  $Ra = 3.10 \times 10^5 - 3.50 \times 10^5$ .

### Kinetic Energy and Nusselt Number

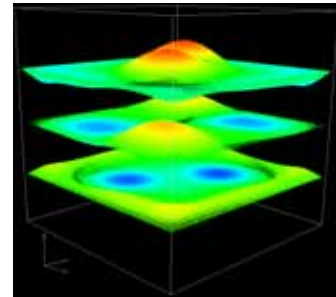
Figure 13 shows a time-mean spatially-averaged kinetic energy  $\bar{K}_{\text{mean}}$  plotted against  $Ra$ , for  $Pr = 7.1$ . And, Fig. 14 shows a time-mean inflow-average Nusselt number  $Nu_{\text{inflow-ave, mean}}$  plotted against  $Ra$ , for  $Pr = 7.1$ . In both Figs. 13 and 14, figures (a) and (b) represent the results for decreasing  $Ra$  and increasing  $Ra$ , respectively.

The increasing rate of the  $\bar{K}_{\text{mean}}$  with increasing  $Ra$  shows a different manner from that of  $Nu_{\text{inflow-ave, mean}}$ , that is, the former is progressive and the latter is asymptotic as  $Ra$  increases.

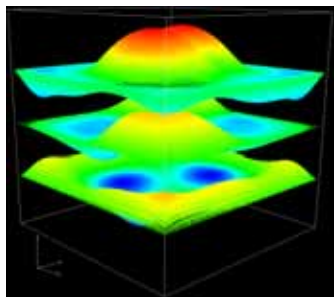




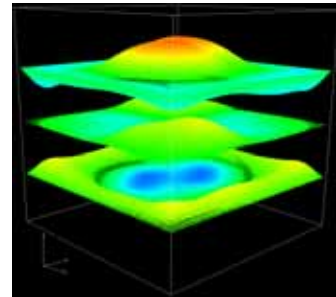
(a) At  $t = 5.016$ : point A in Fig. 8



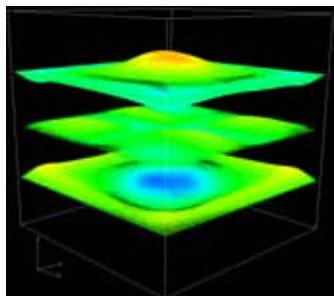
(b) At  $t = 5.022$ : point B in Fig. 8



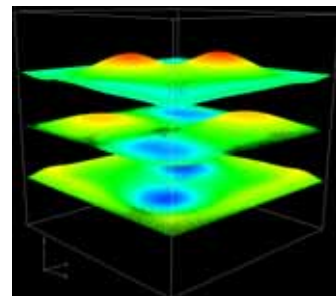
(c) At  $t = 5.026$ : point C in Fig. 8



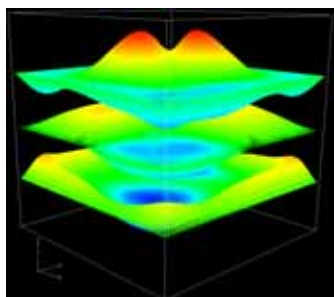
(d) At  $t = 5.030$ : point D in Fig. 8



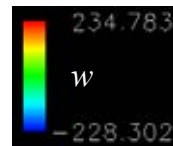
(e) At  $t = 5.034$ : point E in Fig. 8



(f) At  $t = 5.037$ : point F in Fig. 8



(g) At  $t = 5.041$ : point G in Fig. 8

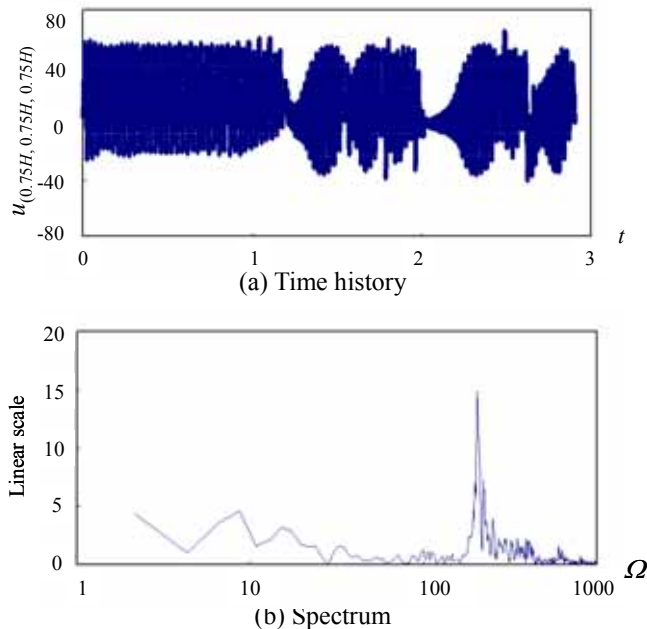


**Fig. 9** Flow structure  $S5_p$ , for  $Ra = 2.65 \times 10^5$  and  $Pr = 7.1$  with increasing  $Ra$  (Perspective views of velocity vectors on the three horizontal plane at  $z = 0.25, 0.5$  and  $0.75$ ). Each arrow's colour denotes the value of  $w$  as shown in a colour bar.

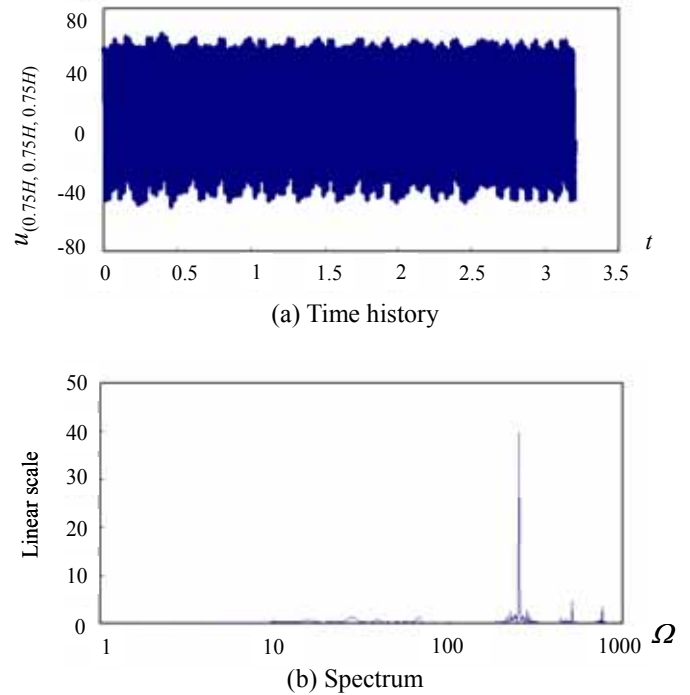
In Figs. 13(b) and 14(b), we can confirm hysteresis effects on  $\overline{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  at  $Ra = 2.28 \times 10^5 - 2.62 \times 10^5$ , respectively. This suggests that the values of  $\overline{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  in the oscillatory convection of the  $S5_p$  tend to be smaller than those in the steady convection of the  $S6$ , respectively. Then, there exist small jumps/drops of  $\overline{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  at  $Ra = Ra_{c2}$ .

## CONCLUSIONS

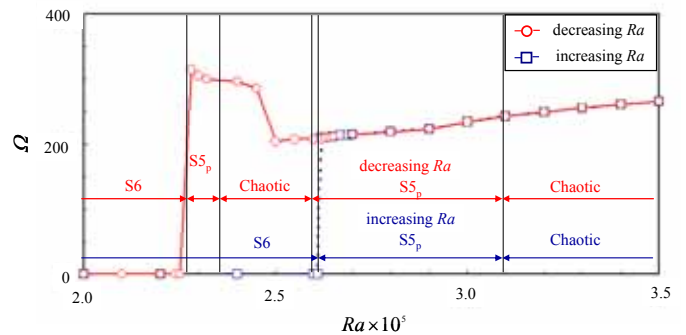
We have specified the second and third critical Rayleigh numbers —  $Ra_{c2}$  and  $Ra_{c3}$ , indicating the transition to oscillatory flow and turbulent flow, respectively. We have found that  $Ra_{c2}$  and  $Ra_{c3}$  are roughly  $2.6 \times 10^5$  and  $3.1 \times 10^5$ , respectively. And, we have observed a hysteresis effect upon the value of  $Ra_{c2}$  with chaotic behaviour at  $Ra \approx Ra_{c2}$ , and revealed flow structures such as  $S2$ ,  $S1$ ,  $S5$ ,  $S6$  and  $S5_p$ . In addition, we investigate the relationship between  $Ra$  and the oscillatory-convection frequency. The  $S5_p$  at  $Ra = 2.28 \times 10^5 - 2.36 \times 10^5$  is not the same as that at  $Ra = 2.62 \times 10^5 - 3.10 \times 10^5$ , from a viewpoint of oscillatory frequency. Accordingly, the Chaotic at  $Ra = 2.36 \times 10^5 - 2.60 \times 10^5$  shows a transition feature between the two different  $S5_p$ . The increasing rate of the  $\overline{K}_{\text{mean}}$  with increasing  $Ra$  shows a different manner from that of  $Nu_{\text{inflow-ave, mean}}$ , that is, the former is progressive and the latter is asymptotic as  $Ra$  increases. Both the values of  $\overline{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  in oscillatory flow tend to be smaller than those in steady flow, respectively. Then, there exist small jumps/drops of  $\overline{K}_{\text{mean}}$  and  $Nu_{\text{inflow-ave, mean}}$  at  $Ra = Ra_{c2}$ .



**Fig. 10** Time history of  $u_{(0.75H, 0.75H, 0.75H)}$  and its spectrum for  $Ra = 2.5 \times 10^5$  and  $Pr = 7.1$  with decreasing  $Ra$  (Chaotic).

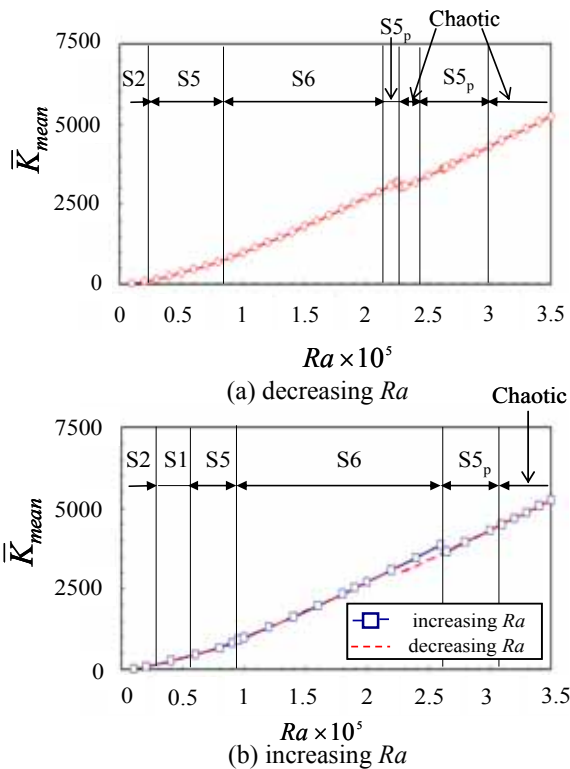


**Fig. 11** Time history of  $u_{(0.75H, 0.75H, 0.75H)}$  and its spectrum for  $Ra = 3.3 \times 10^5$  and  $Pr = 7.1$  with increasing  $Ra$  (Chaotic).

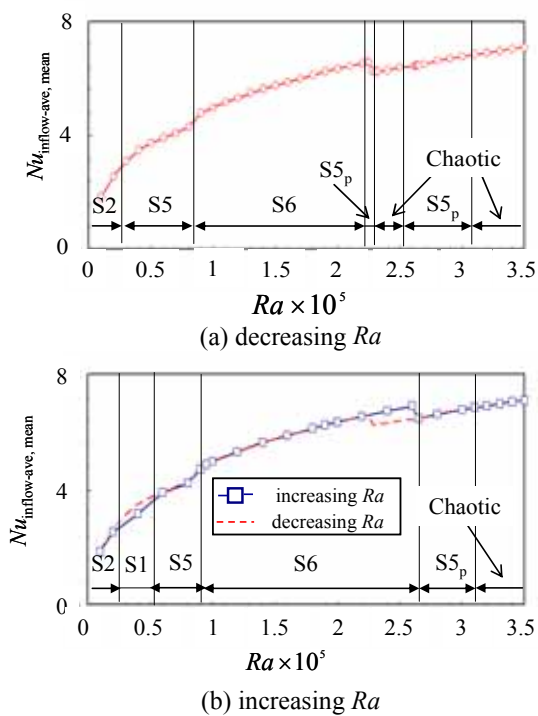


**Fig. 12** Non-dimensional frequency  $\Omega$  of oscillatory convection in  $S5_p$  and Chaotic for  $Ra = 2.0 \times 10^5 - 3.5 \times 10^5$  and  $Pr = 7.1$ .





**Fig. 13 Time-mean spatially-averaged kinetic energy  $K_{\text{mean}}$  against  $Ra$  for  $Pr = 7.1$ .**



**Fig. 14 Time-mean inflow-averaged Nusselt number  $Nu_{\text{inflow-ave, mean}}$  against  $Ra$  for  $Pr = 7.1$ .**

## NOMENCLATURE

$e_z$	unit vector in the $z$ direction	
$g^*$	gravitational acceleration	$[\text{m/s}^2]$
$H^*$	height of a cavity	$[\text{m}]$
$K$	kinetic energy (non-dimensional)	
$M$	mesh size	
$Nu$	Nusselt number	
$p$	pressure (non-dimensional)	
$Pr$	Prandtl number	
$Ra$	Rayleigh number	
$T$	temperature (non-dimensional)	
$t$	time (non-dimensional)	
$\Delta t$	time step (non-dimensional)	
$u, v$	horizontal components of velocity (non-dimensional)	
$\mathbf{u}$	velocity vector (non-dimensional)	
$V$	volume of a cavity (non-dimensional)	
$w$	vertical component of velocity (non-dimensional)	
$x, y$	coordinates in the horizontal directions (non-dimensional)	
$z$	coordinate in the vertical direction (non-dimensional)	

## Greek Letters

$\alpha^*$	thermal diffusivity	$[\text{m}^2/\text{s}]$
$\beta^*$	thermal expansion coefficient	$[1/\text{K}]$
$\varphi$	irreversibility distribution ratio	
$\nu^*$	kinetic viscosity	$[\text{m}^2/\text{s}]$
$\rho^*$	density	$[\text{kg}/\text{m}^3]$
$\Omega$	(angular) frequency (non-dimensional)	

## Superscripts

*	dimensional
-	spatially-averaged

## Subscripts

c	on cold wall
h	on hot wall
inflow-ave	inflow-average
local	local
mean	time-mean

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