

THESIS

INVESTIGATION OF SUPERTURBOCHARGER PERFORMANCE  
IMPROVEMENTS THROUGH STEADY STATE ENGINE SIMULATION

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## ABSTRACT

### INVESTIGATION OF SUPERTURBOCHARGER PERFORMANCE IMPROVEMENTS THROUGH STEADY STATE ENGINE SIMULATION

An integrated supercharger/turbocharger (SuperTurbo) is a device that combines the advantages of a supercharging, turbocharging and turbocompounding while eliminating some of their individual disadvantages. High boost, turbocompounding, and advanced controls are important strategies in meeting impending fuel economy requirements. High boost increases engine power output while many losses remain constant, producing an overall efficiency gain. Turbocompounding increases engine efficiency by capturing excess exhaust turbine power at high speed and torque. Supercharging increases low speed high torque operating performance.

Steady state performance gains of a Superturbocharger equipped engine are investigated using engine simulation software. The engine simulation software uses a 1-D wave flow assumption to model the engine's unsteady flow behavior through one dimensional pipes. With these pipes connected to other engine components the overall performance of the engine can be modeled. GT-Power was chosen to run the simulations due to an already correlated engine model being available. This software is used to 'tune' an existing stock engine model to approximate stock engine data over the full speed and

torque range. The SuperTurbo is added to the model and simulations are performed over the full engine speed and torque range for direct comparison with the stock engine.

The model results show turbocompounding to be most effective at high speeds and torques in the area above 10 bar BMEP in the 3000 – 4000 RPM range and above 5 bar BMEP in the 500 – 6000 RPM range. In addition to turbocompounding there are fuel savings due to the reduced use of the compressor when it is not needed. With the stock configuration there is boost pressure created by compressor power that is then restricted by the throttle in the 2500 RPM range in the 8-12 bar BMEP range on up to 6000 RPM in the 2-10 bar BMEP range. The control of compressor speed to produce no boost at these locations improves efficiency by not wasting energy creating boost that is not needed.

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# CHAPTER 1: INTRODUCTION

## 1.1 Scope and Focus of Thesis

In this work, an already calibrated engine model is modified by adding a model of a device called a superturbocharger (SuperTurbo). This device incorporates a transmission between the turbocharger shaft and the engine's crankshaft. The transmission consists of a high speed traction drive section and a continuously variable section that will allow control of turbocharger speed, independent of crankshaft speed. This allows for more options for load control as the load can now be adjusted with both the SuperTurbo gear ratio and the throttle valve.

Chapter one discusses work leading up to this investigation as well as the current status of the SuperTurbo development. Chapter two describes the modeling software and the available calibrated stock engine model, as well as the SuperTurbo addition. Chapter three describes the simulations of the two independent load control mechanisms consisting of the throttle and the SuperTurbo. Chapter four summarizes the results, presents a conclusion and possibilities for future work.

## 1.2 Literature Review

There are several aspects of engines that relate to current efforts to make them more fuel efficient, while keeping emissions low and maintaining drivability. Both steady state and transient modes are considered. Forced induction as a means of downsizing the engine is

investigated. Waste heat recovery methods can be used to increase fuel efficiency. Control efforts are also an important consideration.

### **1.2.1 Transient and Steady State Performance**

It has become increasingly common for smaller, better performing engines to replace larger engines in automobiles. In the past, engine manufacturers were able to accommodate increased performance requirements with increased displacement. However, since the advent of political uncertainty surrounding emissions and fuel prices, it has become necessary for engine and automobile manufacturers to continually improve fuel economy and lower emissions without sacrificing drivability. A common goal has been to produce equivalent power, through means such as boosting, from a smaller engine leading to potential fuel and emissions savings <sup>(1), (2), (3), (4)</sup>. This technique has often been referred to as engine downsizing <sup>(1), (5), (6)</sup>. Through the study of previous and current efforts along with modern engine modeling software, further engine downsizing can be realized.

Engine performance relating to fuel economy, emissions, and power can be divided into two operating modes: transient and steady state. Steady state behavior has usually been characterized by low load constant speed driving, such as on the highway. In stationary applications where engines run at near constant speed and load, the engines can be designed around a single operating point, making them among the most efficient of all reciprocating engines available. The problem arises with trying to obtain improved fuel economy while at the same time having satisfactory transient performance.

A vehicle can experience several different types of transients. One type of transient response is when the speed remains relatively constant while the load is changed, which occurs as the clutch is being let out on an automobile with a manual transmission. Another type of

transient situation is where the load is kept relatively constant while the speed undergoes transient behavior. In urban settings automobiles most often undergo a combination of the two where load and speed are both experiencing transient behavior <sup>(7), (8)</sup>. This is where drivability of the downsized engine comes into play. The automobile will intermittently need extra power for the quick accelerations necessary in daily driving. If the downsized engine is not able to adequately accept its given load at a given speed, the engine speed will drop and in the extreme case, eventually stall if corrective measures such as downshifting are not implemented <sup>(8)</sup>.

When a torque (or horsepower) curve vs. engine speed is given for a specific engine, it indicates the steady state torque at a given engine speed. Loading (increased torque) the engine above this line results in a decrease in engine speed. Steady state loads are associated with friction in the system as well as air drag forces. The transient situation must also take into account the energy used to accelerate the inertial loads <sup>(8), (9), (10)</sup>. As a car is accelerating through its gear range, it is continually going through the speed range of the engine. This requires the engine to overcome inertia from the engine, the drive train, and from the automobile itself, in addition to the friction and air resistance that is also experienced at steady state operation. This plays an important role in the drivability of an automobile.

Another disadvantage during transient performance is the rapidly changing intake air. In contrast with steady state operation, where there is ample time for the ECU to optimize the air fuel ratio accurately, transient response is susceptible to transient response characteristics in all control systems, such as overshoot and settling time, resulting in poorer fuel and emissions ratings <sup>(7)</sup>. When an automobile engine operates at steady state mode, it has the advantage of being able to accurately keep the air to fuel (A/F) ratio at the desired level with an O<sub>2</sub> sensor due to the near constant rate of air being delivered. Along with the use of a 3-way catalyst, satisfactory power, fuel consumption, and emissions can be realized.

Automobile engines require a large range of operating conditions that need to be considered for overall performance. Engines must have satisfactory steady state fuel and emissions performance and maintain this performance during transient operation, which require dramatic increases in load. These considerations must also take into account the overall economic value with respect to the additional complexity. Boosted engines are becoming increasingly more complex, but the cost benefit ratio of such systems has been realized through their ever-increasing use.

### **1.2.2 Forced Induction**

As the saying goes, “There is no replacement for displacement.” A forced induction engine is simply not capable of meeting the transient performance of a naturally aspirated engine of the same power output <sup>(3), (4), (8), (9)</sup>. This has been the main challenge confronting the use of forced induction engines with regards to both superchargers and turbochargers.

Forced induction refers to the concept of forcing more air into the intake manifold stream than a naturally aspirated engine is capable of. This compressed air is referred to as boost. The two most common types of forced induction are supercharging and turbocharging. While turbocharging is sometimes referred to as a type of supercharging <sup>(1)</sup>, they will be referred to as two separate concepts herein. Supercharging uses energy from the crankshaft to drive a compressor; turbochargers use exhaust gas energy to drive a compressor. Another type of forced induction is the Comprex pressure wave charger <sup>(9), (10), (11)</sup>. This system uses the exhaust gas pulses as a means for compressing the intake air. There are also studies on injecting a stored source of compressed air into the intake stream <sup>(8)</sup>.

Supercharging is historically associated with high performance and racing applications without regard for fuel economy. Compared to other pressure charging methods, supercharging

is not very desirable for steady state performance as it requires crankshaft power. Additionally, at steady state operation this extra power is not needed. Through the technique of downsizing, a supercharged engine can replace a larger naturally aspirated engine for transient needs while still having adequate low load capability for steady state operation. A supercharger needs to be selected that can quickly develop boost with minimal losses at low loads when not in use.

In one study, a positive displacement backflow vane type supercharger was selected for its capability of developing the necessary boost at speeds close to engine speed due to its closely matched volumetric efficiency<sup>(1)</sup>. This eliminated the complexity of high speed operation. It was also chosen since it was compact and lightweight compared to compressors with similar outputs. With the proper selection of a supercharger, the engine acts as a smaller naturally aspirated engine at steady state speeds while still being able to develop the torque needed at higher loads with the assistance of the supercharger. While good results were obtained, this design was not widely implemented. Figure 1.1 shows fuel economy and 0-60 mph time

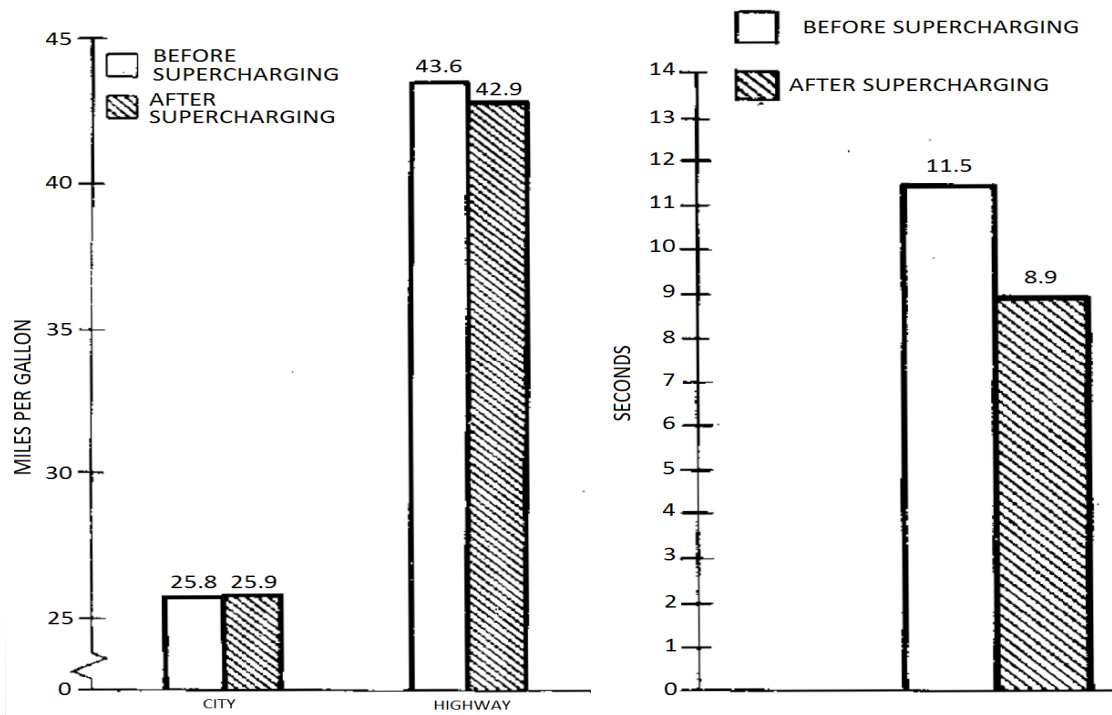


Figure 1.1: 1980 Scirocco fuel economy and 0-60 mph times<sup>(1)</sup>

comparisons for a 1980 Volkswagen Scirocco with a 1.6 liter inline 4 cylinder engine with a 5 speed manual transmission. The axle ratio was kept constant at 3.89:1 and the continuous fuel injection system left unchanged. The graph shows almost identical city fuel economy with slightly worse highway fuel economy, but a 0-60 mph time decrease from 11.5 to 8.9 seconds (23% decrease). Figure 1.2 shows the same comparison for a 1979 Dodge Omni with a 1.7 liter

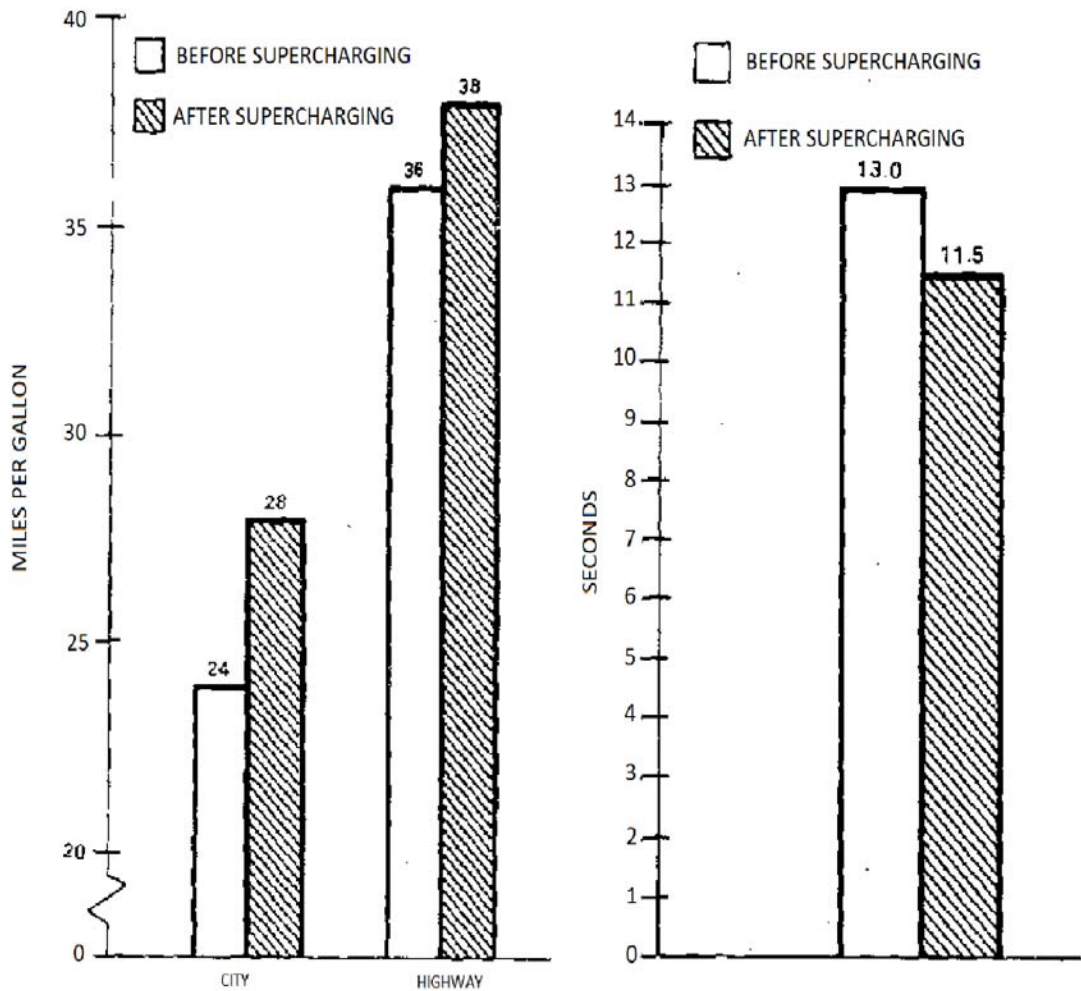


Figure 1.2: 1979 Omni fuel economy and 0-60 mph times <sup>(1)</sup>

inline 4 cylinder engine with a 4 speed manual transmission. The axle ratio was 3.47:1 as received and reduced to 3.08:1 for the supercharged testing. The fuel system was switched from a Holley 2v to a Pierburg 1v most likely in order to be able to operate under boost pressure. The

results show improved fuel economy for both city and highway (about 17% and 9% respectively), as well as improved 0-60 mph time response (13% faster).

Turbochargers are becoming increasingly used for improved efficiency. Unlike the supercharger, a turbine in the exhaust system powers the compressor instead of the crankshaft. The losses due to the increased back pressure in the exhaust are overcome, by the advantages of the forced induction. This makes it ideal for improved load acceptance during steady state applications where the engine performance can be designed around a smaller range of operating points. However, turbochargers have inherent disadvantages with transient performance.

In a turbocharger, the compressor and turbine must be matched for a required pressure ratio and mass flow. Perfect matching can only be accomplished for one particular operating point<sup>(12)</sup>. Due to this, compromises in efficiency are often made in order to enable operation over a wider range of conditions.

When a turbocharged engine increases its speed, the excess turbine power needed to accelerate the turbo machinery inertia can result in what is known as turbo lag, a temporary mismatch of the turbocharger-engine system. This causes reduced fuel efficiency and higher emissions during this period. A turbocharger that is optimized for steady state performance will have a considerable amount of turbo lag. To address this issue it is common practice to sacrifice efficiency even further in order to reduce the inertia and thus the turbo lag<sup>(12)</sup>.

Boosted engines also have the disadvantage of an increased likelihood of self ignition, commonly known as knock. This is often dealt with by a retarded spark advance. Turbocharged engines have yet another disadvantage involving exhaust gas temperatures. The maximum turbine inlet temperature can easily be exceeded during full load operation, requiring the engine to run rich in order to reduce turbine inlet temperature to an acceptable level, as well to



decrease the tendency for knock by reducing in-cylinder temperature. Periods of transient response, where the engine is required to run rich, leads to higher fuel consumption and emissions<sup>(5)</sup>.

Testing was done to compare the differences between a turbocharger and a supercharger on a 2.3 liter spark ignition engine<sup>(4)</sup>. The first objective was to optimize the turbocharger operation in two areas. First, tests were conducted to evaluate the effect of turbine inlet temperature on turbocharger performance. The stock manifold used was excessively long resulting in a temperature loss of about 56 C compared to the rerouted crossover pipe, which was 60% shorter. The shortened pipe had a significant impact in turbocharger speed change with an improvement of about 31%, while steady state tests showed only a slight improvement. Next, the effect of intake volume between the compressor discharge and the ends of the intake manifold were investigated. Through calculations it was predicted that a 25-fold increase in volume (from 40 to 1000 cu. in.) would affect the response time by 0.1 seconds. These surprising results were then verified experimentally where an increase from 50.6 cu. in. to 1174 cu. in. resulted in a delay in the range of 0.1 to 0.125 seconds. Following the turbocharger optimization tests were a series of tests to compare the supercharger setup with the turbocharger setup. The supercharger used was the Bendix clutched, positive displacement type referenced earlier. The transient tests conducted consist of transient load only, from an idle condition at a set speed to wide-open throttle at that same speed. The results are shown in Figure 1.3. The supercharger shows a quick response in the time it takes to develop boost, where the turbocharger comes up to ambient pressure quickly as the throttle is opened, but then develops boost at a much slower rate taking up to a second longer at 3000 RPM and never reaching full boost at the 2000 RPM condition. The slow turbocharger boost rise in the 2000 RPM condition triggered the ignition timing switch, which

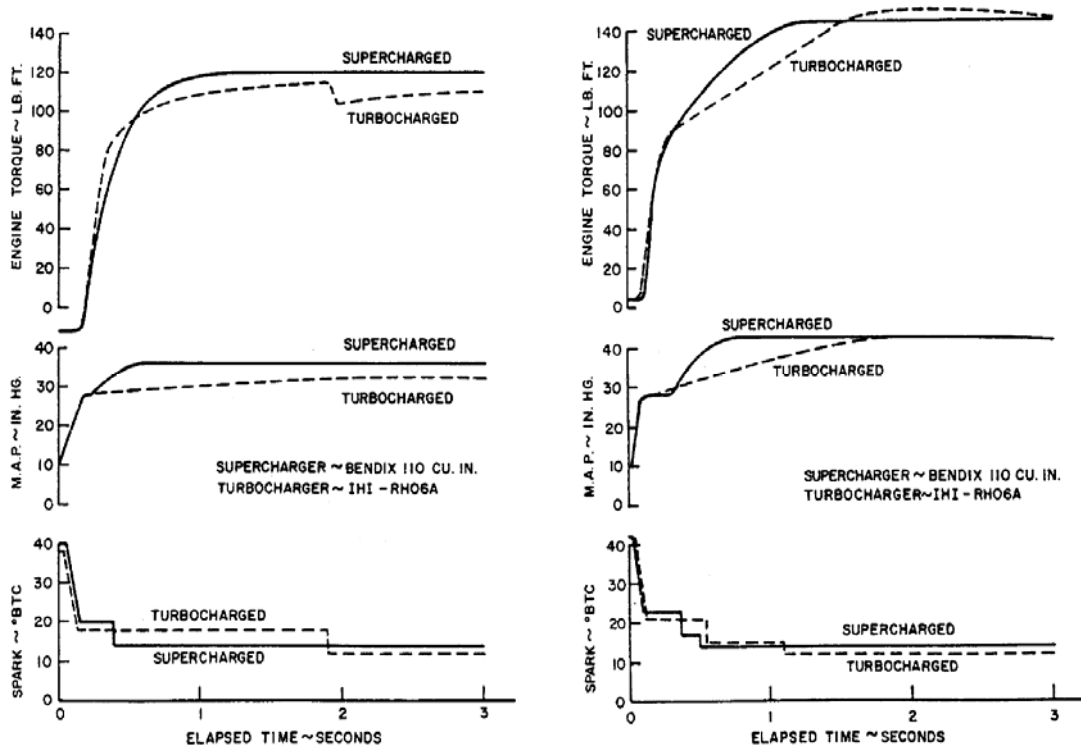


Figure 1.3: Comparison of turbocharged/supercharged 2.3l EFI spark-ignited engine performance of WOT transient at 2000 rpm (left) and 3000 rpm (right) <sup>(4)</sup> resulted in 6 degrees of spark retard and added to the torque difference as shown. The steady state performance tests are shown in Figure 1.4.

The major differences between supercharging and turbocharging appear to be the higher temperature difference across the supercharger compressor indicating more parasitic losses, whereas the turbine in the turbocharger has a much higher level of backpressure in the exhaust, which also leads to pumping inefficiencies. Another tradeoff between these two setups was that the supercharger was capable of accepting load more quickly while the turbocharger was able to accept more load overall.

### 1.2.3 Waste Heat Recovery

For reciprocating engines there is a rule of thirds which approximately states that only one third of the combustion energy is used as mechanical energy with another third of the

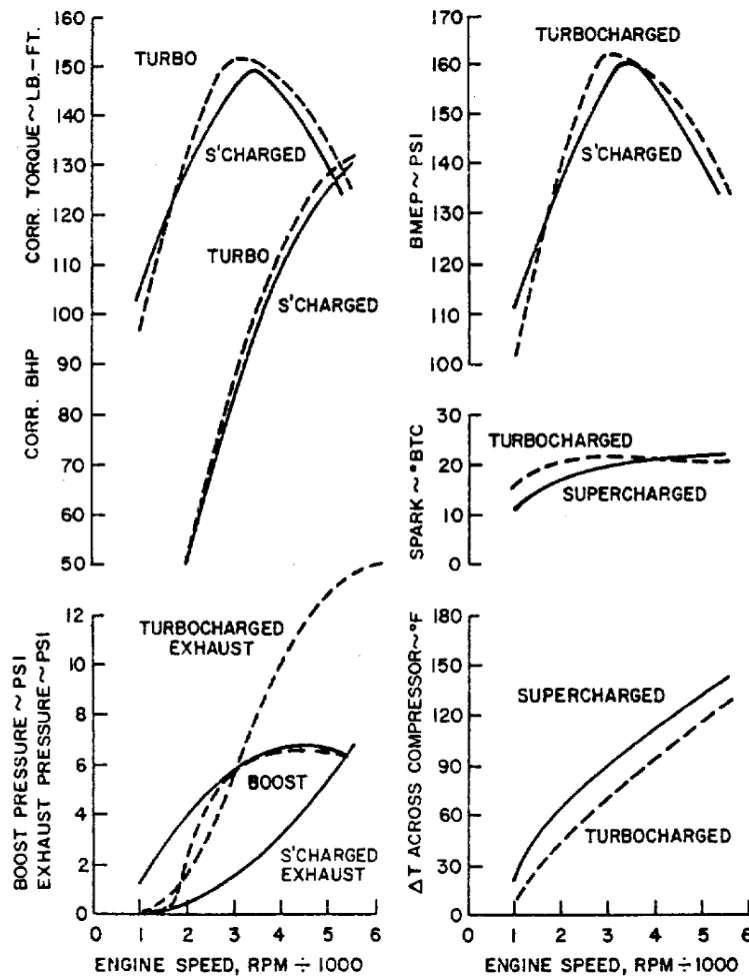


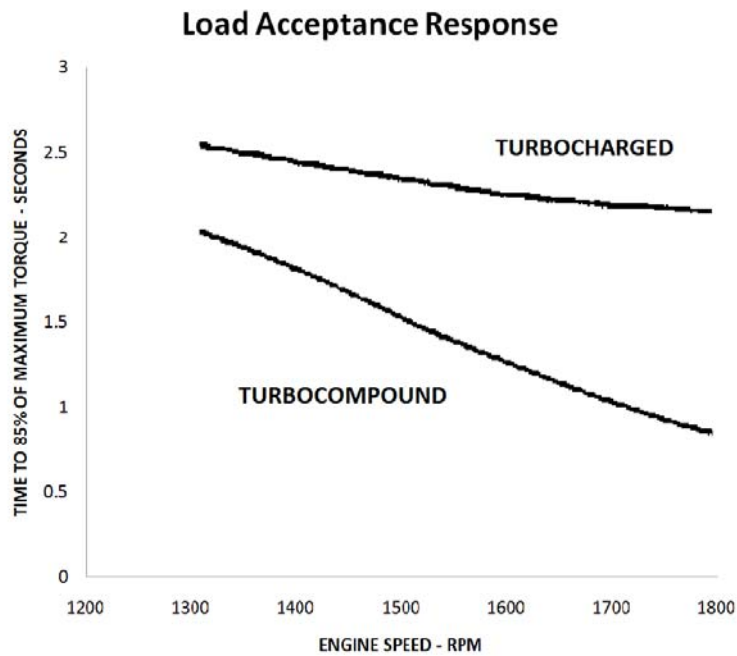
Figure 1.4: Steady state WOT performance tests <sup>(4)</sup>

considerably more energy available to a turbine than a compressor is capable of using at steady state. This leads to the method of turbocompounding where extra energy available in the exhaust that is not used for turbocharging is converted to mechanical energy with a power turbine and then transferred to the crankshaft of the reciprocating engine.

Mechanical turbocompounding has an extensive history dating back to WWII with a wide range in application and arrangement <sup>(13), (14)</sup>. While turbocompounding might commonly be thought of as a solution to improved steady state operation, it has been shown that if the turbines are placed in series, the compressor turbine is able to maintain higher speeds during transient load tests, which in turn improves the transient response of the system <sup>(15)</sup> as shown in

energy transferred to the cooling system and the final third wasted in exhaust heat. While modern engines are more efficient thermal energy in the exhaust still accounts for a significant amount of the total thermal energy released during combustion. A turbocharger is capable of converting some of this energy into compressed air, but there is

Figure 1.5. This figure shows the turbocompounded engine reaching 85% of maximum load from



one half to over a second faster across the engine speed range shown.

Both electrical and mechanical turbocompounding have been studied<sup>(12), (16)</sup>. Mechanical turbocompounding seems to be the

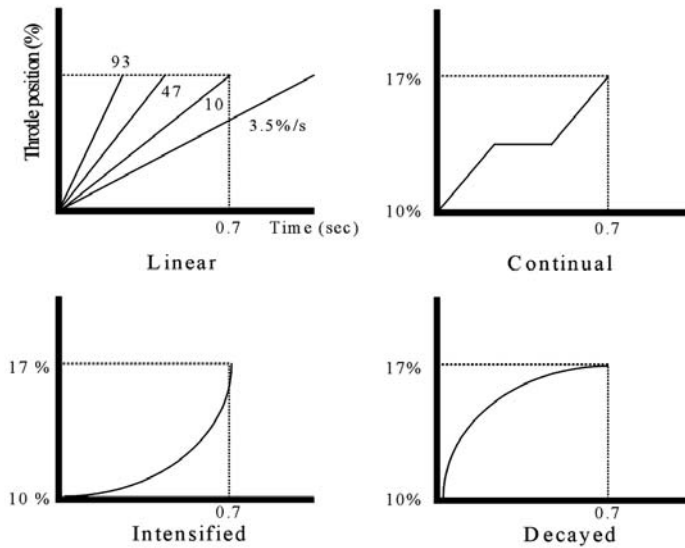
**Figure 1.5: Comparison of load acceptance response between<sup>(15)</sup>** the electrical system’s increased inertia and its sensitivity to the high temperatures inherent in the turbocharger system<sup>(17)</sup>.

Other methods studied for recovering waste heat energy include Rankine bottoming cycles<sup>(16)</sup> and charge air cooling<sup>(18)</sup>. Rankine bottoming cycles use exhaust heat to create a steam cycle that goes through a power turbine. Charge air cooling uses exhaust gas energy in combination with heat exchangers, suction compressors, and expanders to cool the intake air.

### 1.2.4 Control Strategies

With the increased complexity of modern engines due to downsizing techniques, it is becoming necessary for controls to play a larger role in engine performance. Modern fuel systems are able to deliver fuel at a very controllable rate. The more difficult part of air fuel ratio control deals with precise control of the intake air.

Although there has been much research going on in the area of transient engine system control, it helps to first have an accurate and quantified analysis and evaluation of the transient behavior. This has led to the development of transient response specifications as well as the classification and evaluation of different acceleration types as shown in Figure 1.6. These



correspond to throttle changes from 10-17% assuming 0% is fully closed and 100% is fully open. The speeds were changed at a constant load setting. It was shown that the driving behavior of an individual can also have an effect on performance.

Figure 1.7: Acceleration speeds and types <sup>(7)</sup>

Figure 1.7 shows the

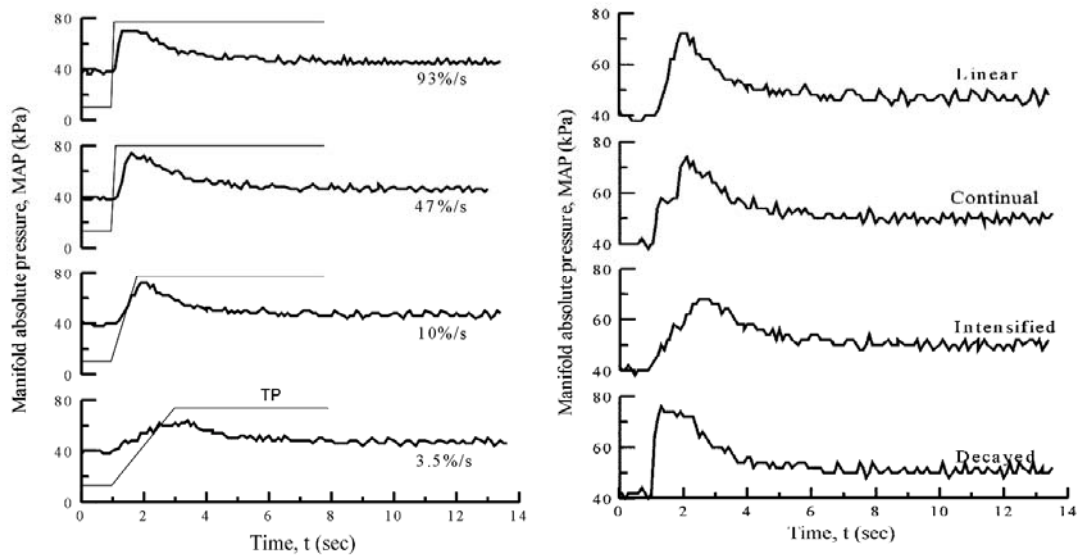
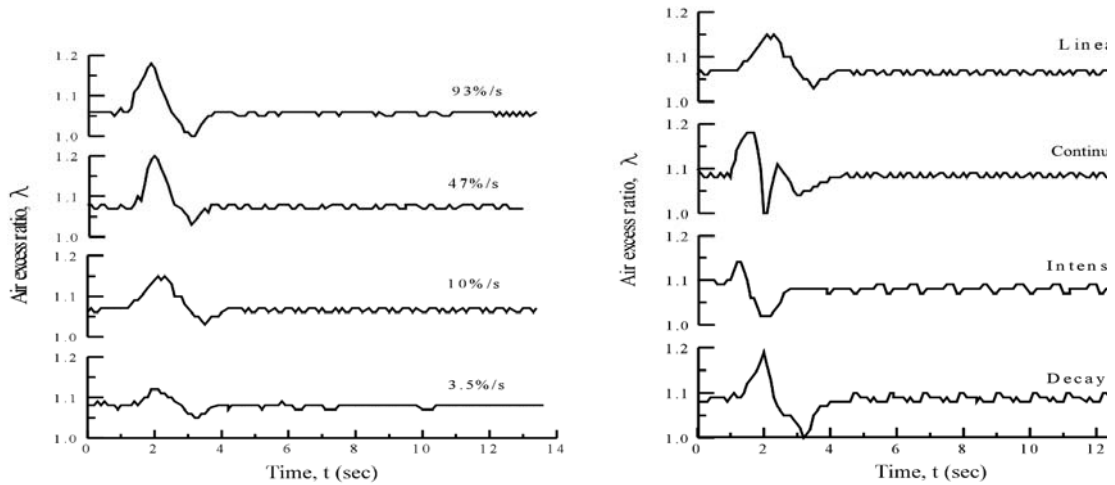


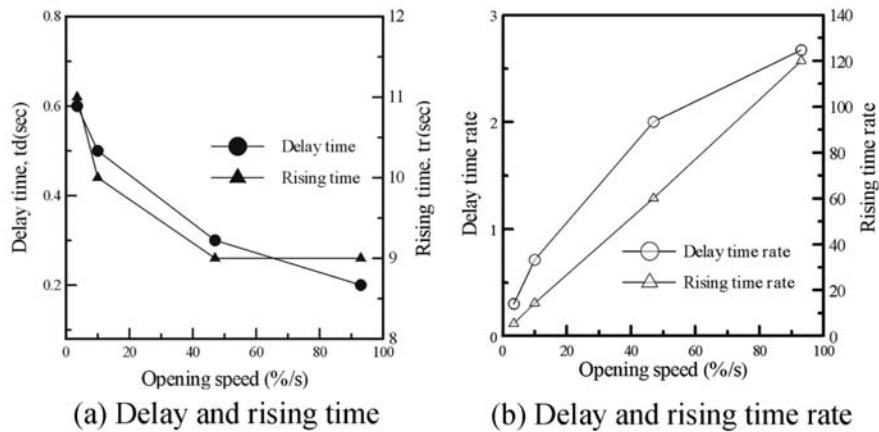
Figure 1.6: MAP responses according to the throttle valve opening speed (left) and acceleration type <sup>(7)</sup>

difference in manifold absolute pressure (MAP) vs. time for the different acceleration types. The air excess ratio of an automobile<sup>(7)</sup> was also affected as shown in Figure 1.8, which in turn leads



**Figure 1.8: Air excess ratio responses according to the valve opening speed and acceleration type<sup>(7)</sup>**

to poorer emissions. The engine speed response for all acceleration types except for the two slowest linear types were shown to be fairly similar indicating that there is a limit to acceleration performance as shown in Figure 1.9, and anything beyond that limit only leads to poorer fuel



**Figure 1.9: Transient response specifications of RPM<sup>(7)</sup>**

from pedal position, could be a desired control strategy for improved fuel and emissions performance.

economy and emissions. This lead to the idea that control of throttle position, independent

Integration of throttle and wastegate control for turbocharged engines has been studied<sup>(3)</sup>. Due to the increased need for accurate throttle control with turbine engines, control strategies have been developed for electronically actuated intake throttles for improved response. The airflow can then be controlled independently of the foot pedal position. With this setup, the driver still requests a speed or load increase, but the control of the intake throttle valve are performed by the computer controlling the overall system.

Decoupling the throttle response from the foot pedal has led the way to additional improvements in air flow control. Modeling performed on an engine using GT-Power engine simulation software shows the effect of different throttle arrangements for a supercharger and compares them to the effects of different fueling and wastegate management for a turbocharger<sup>(5)</sup>. The engine simulations were first calibrated according to experiments involving spark advance, combustion, max combustion pressure, full-load curves, etc. Once the engine model was validated, changes to the model involving the location of the throttle were implemented. Intake air can be controlled by many different throttle arrangements for a supercharged engine. The throttle can be placed either before or after the compressor. There was also the option of having a bypass/recirculating valve for the compressor on both of the previous conditions for a total of 4 different arrangements. Modeling on 3 of the 4 configurations, leaving out throttle after the compressor with bypass, showed that throttle before the compressor, with bypass, has the best low load fuel consumption with the same version without the bypass slightly higher as shown in Figure 1.10. The condition with the throttle after the compressor with no bypass was shown to be much less efficient at these operating points, largely due to the operating points being at high compression ratios lying outside the original compressor map which only went up to a pressure ratio of 3. The effect of having a bypass with the throttle after the compressor was not shown. The use of bypass in this

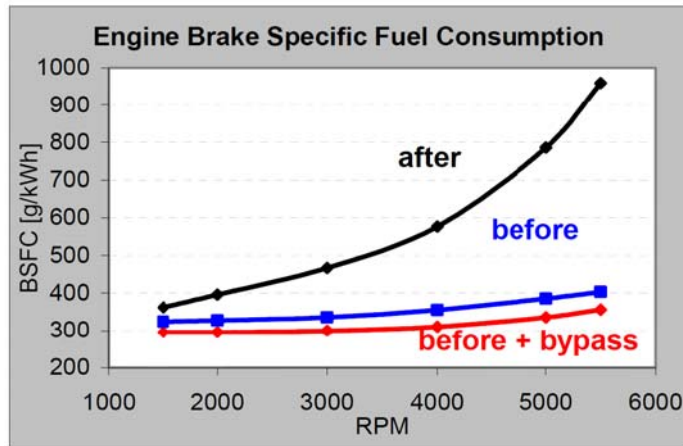


Figure 1.10: Comparison of fuel consumptions for three types of supercharger regulation <sup>(5)</sup>

situation could have resulted in lower pressure ratios that are inside the compressor map. This engine was compared to a turbocharged version with different variations in fuel control and wastegate control.

The turbocharged version was shown to produce more power

at reduced fuel consumption over the full load range at maximum turbo speed. The first condition was the maximum allowable torque and power with the disadvantage of poorest fuel economy. The second condition was modified by reducing fuel use at lower engine speeds, only so that the high speed power and fuel consumption is the same. The third version was the economic version where the wastegate diameter was enlarged causing both power and fuel consumption to drop. The first situation had the wastegate open during low engine speeds on the full load curve. A more optimum situation would be that the wastegate is closed at lower engine speeds and then opened at the higher engine speeds. These results lead to the idea that a larger turbocharger could reduce fuel consumption and increase efficiency <sup>(5)</sup>.

Another valve control approach consists of a valve system that closes off the intake to the compressor and bypasses it at low loads for a turbocharged engine, labeled as the fast response turbocharging method. This testing was done empirically due to unavailability of turbine maps and zero flow compressor maps. Figure 1.11 shows that by having zero flow through the compressor, the speed of the compressor was substantially higher at a lower load than the WOT compressor speed. This implied the possibility of improved transient response <sup>(2)</sup>.



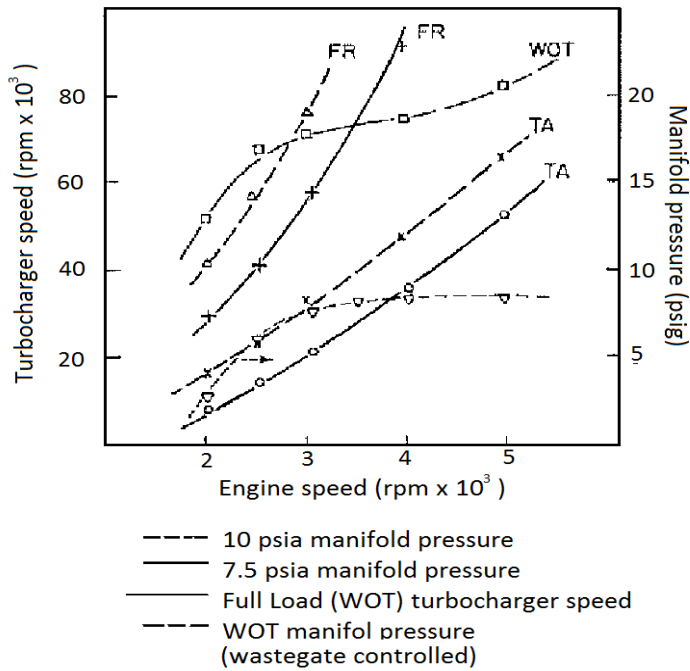


Figure 1.12: Turbocharger speed for conventional system (TA) and for zero airflow (FR) for two sub-ambient pressures<sup>(2)</sup>

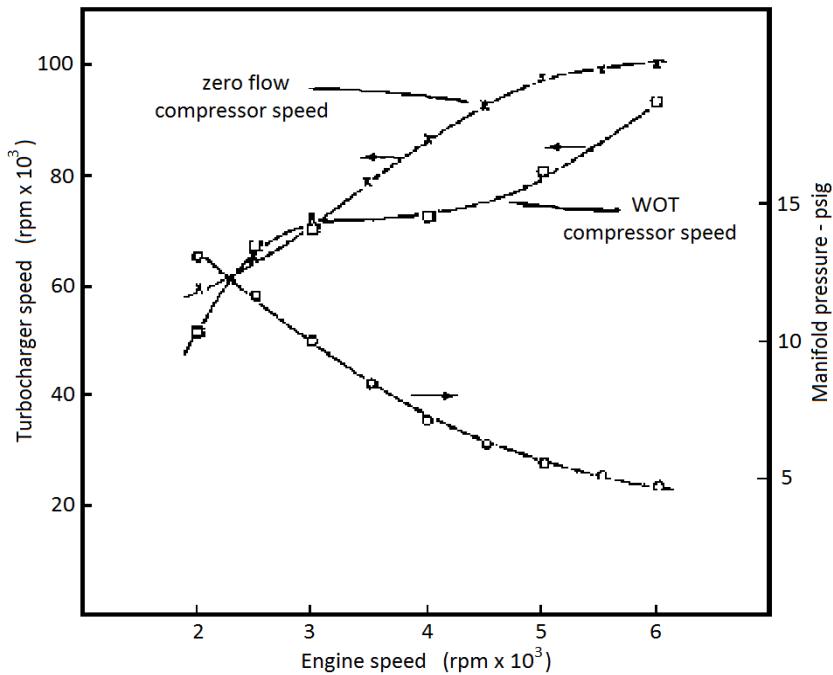


Figure 1.11: Control of zero flow compressor speed for fast response system<sup>(2)</sup>

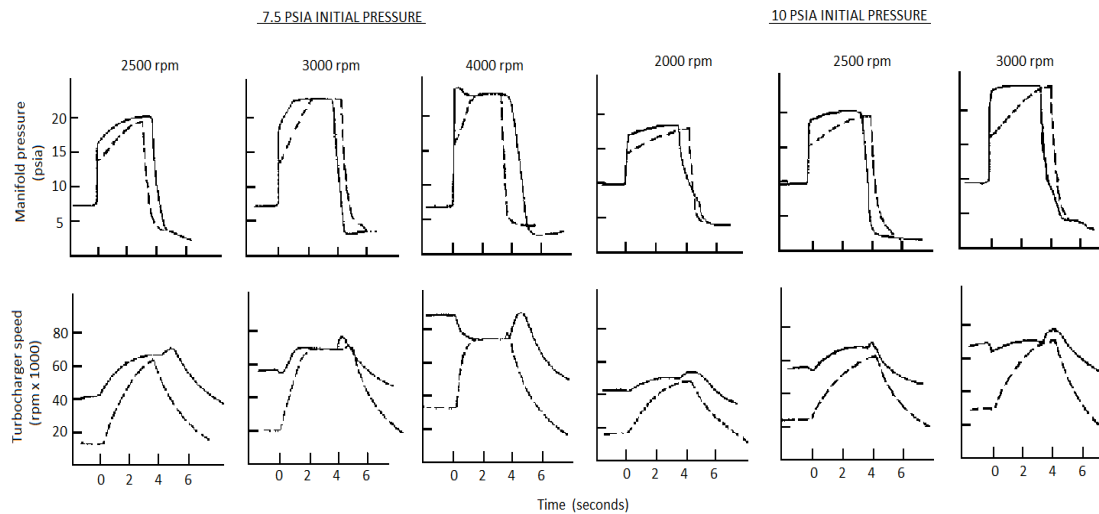
After this finding, the turbo shaft speed was limited by the use of a throttle that choked the bypassing airflow through the engine to a level low enough that the turbine had only enough power to keep the compressor speed at an acceptable level, as shown in Figure 1.12. It was also determined that bearing

losses in the compressor dominated at low speeds while turbulent losses dominated at higher speeds, which helps to keep the compressor from spinning too fast.

Other methods

such as use of a wastegate control to maintain shaft speed were also found to be effective, but not essential.

After the effective valve areas were chosen in a manner to ensure smooth airflow transfer, transient tests were implemented. Tests were carried out where the engine speed was held constant at a low load and then the throttle was opened suddenly as shown in Figure 1.13.



**Figure 1.13: Transient response of conventional (- -) and fast response (-) system<sup>(2)</sup>**

It was found that when initial low load turbo speed was below steady state full load speed, manifold pressure immediately following fast throttle opening was above ambient pressure but less than full load. If the initial low load turbo speed was above steady state full load speed, the manifold pressure would temporarily exceed steady state full load pressure while the turbo speed decelerated. When installed in the same automobile, the fast response system showed significant improvement over the traditional system as shown in Table 1.1. The fast response system also allowed the turbine shaft to maintain its speed during gear changes. Some drawbacks to this system were the complexity of the mechanical valve linkages and the possibility of surge instability, although in the experiments carried out, no evidence of surge was found. There were additional measures suggested that could help suppress surge oscillations if

**Table 1.1: Level road acceleration tests <sup>(2)</sup>**

	<u>1</u>	<u>2</u>	<u>3</u>	<u>4</u>
GEAR	3	4	4	3
ENGINE SPEED (RPM)	3000	2500	2800	3600
INITIAL ROAD SPEED (MPH)	45	53	60	55
FINAL ROAD SPEED (MPH)	70	70	80	70
INITIAL MANIFOLD PRESSURE (psia)	10	10	7.5	7.5
$\Delta T$ (sec) CONVENTIONAL	5.6	6.1	7.1	3.3
$\Delta T$ (sec) FAST RESPONSE	5.1	5.1	6.3	2.8

present. Overall, the fast response system was shown to improve performance over a conventional turbocharging system <sup>(2)</sup>.

Air cycle calculations were carried out to investigate several load control strategies as alternatives to throttling for an Otto cycle engine <sup>(19)</sup>. This study was motivated by the fact that Otto cycle engines are not capable of the part load efficiencies of a diesel engine

due primarily to throttling losses. After presenting the base case of a throttled engine (strategy I), several alternatives were presented consisting of variable fueling, variable compression ratio, and early intake valve closing. After studying the tradeoffs between these strategies, the final two strategies for comparison consisted of versions of variable valve timing with boost. The first strategy met load demand first with boost, and if no boost was available, would utilize delayed intake valve closing (strategy V-I). The second concept delayed valve timing first, then used turbocharger boost to meet the high load demands (strategy V-II). The results indicated that the first method had increased thermal efficiency and reduced peak pressure over the second method as shown in Figure 1.14. Then when compared to the naturally aspirated throttled engine, it was shown that there are efficiency gains over the full range of loads, as well as peak pressure reductions over much of the range as shown in Figure 1.15.

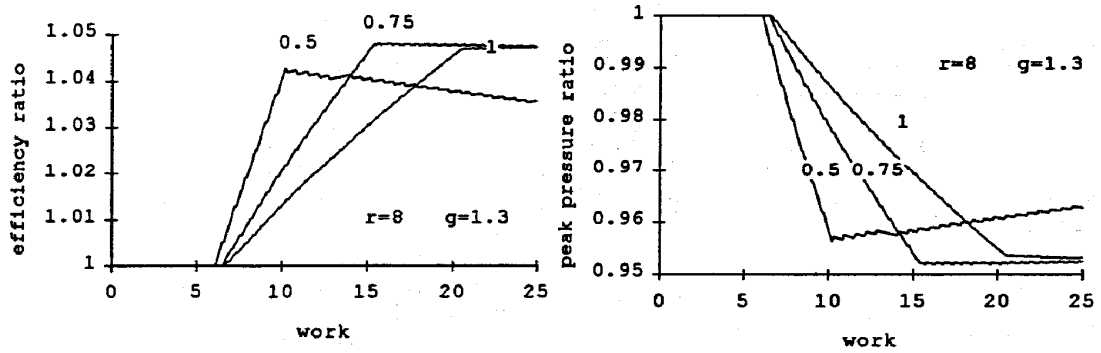


Figure 1.14: Ratio of efficiency of strategy V-I to strategy V-II for different values of F (left) and ratio of peak pressure of strategy V-I to strategy V-II for different values of F (right)<sup>(19)</sup>

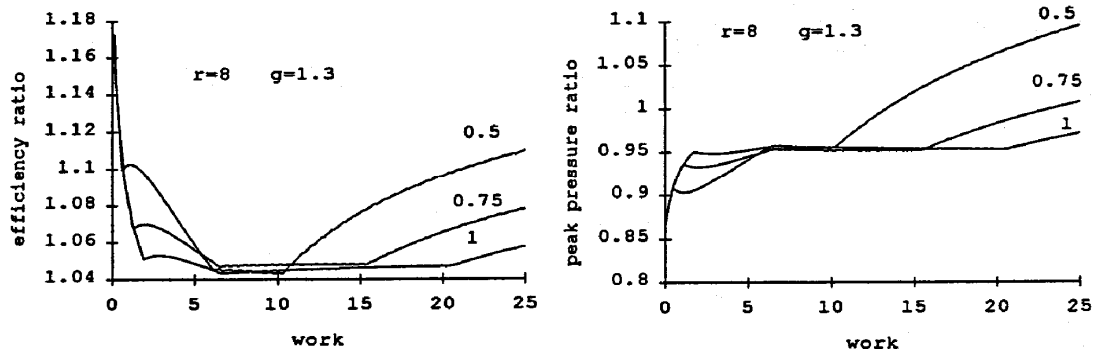


Figure 1.15: Ratio of efficiency of strategy V-I to strategy I for different values of F (left) and ratio of peak pressure of strategy V-I to strategy I for different values of F (right)<sup>(19)</sup>

### 1.3 SuperTurbo

The SuperTurbocharger is a concept currently being developed by VanDyne SuperTurbo. As shown in Figure 1.16, it allows power to flow from the crankshaft to the turbine shaft to supplement turbine power when substantial boost is needed and it also allows turbocompounding where excess turbine power flows back to the crankshaft. As Figure 1.17 illustrates it combines the advantages of a

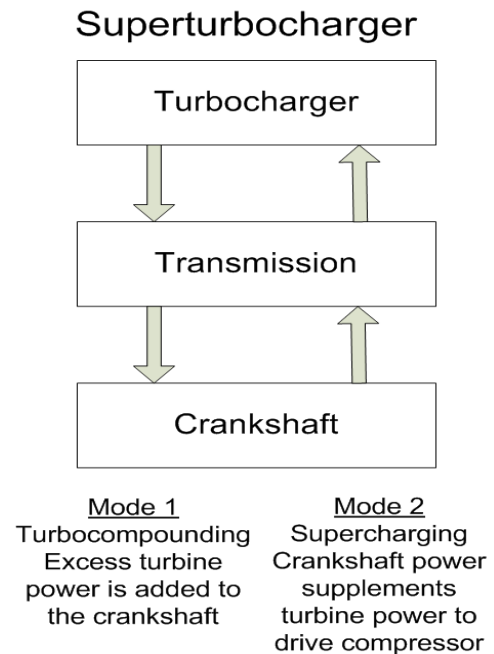
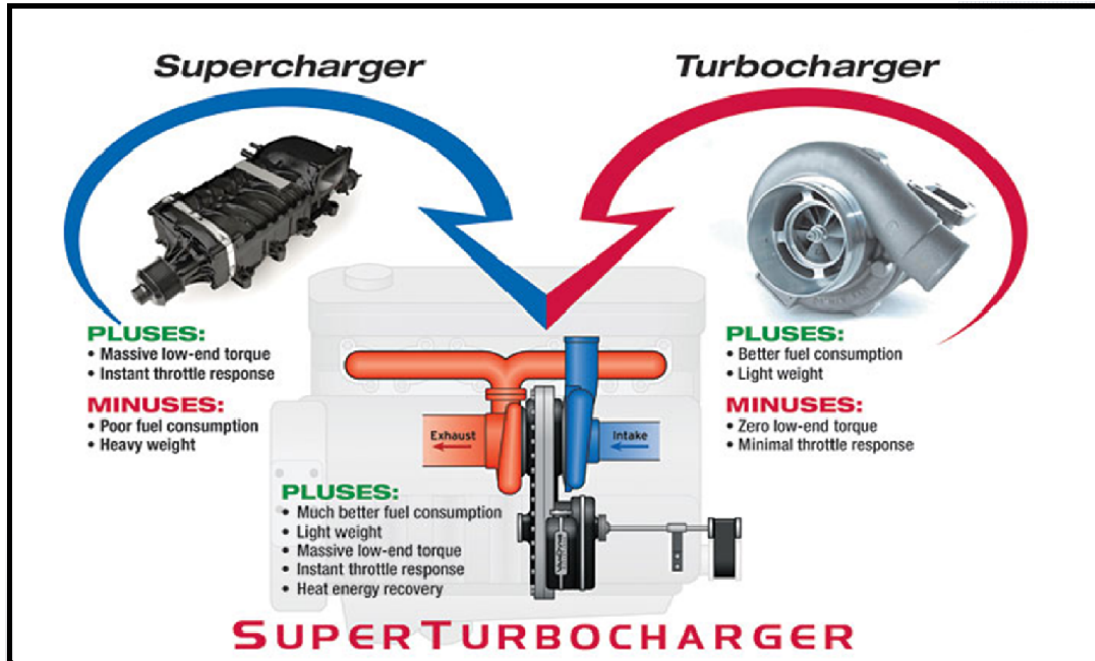


Figure 1.16: SuperTurbo power flow diagram

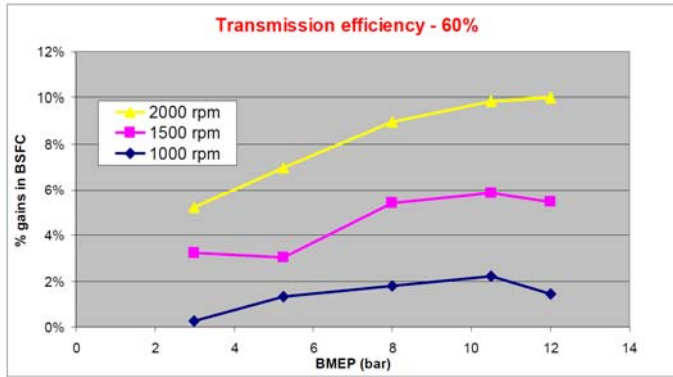


**Figure 1.17: The SuperTurbocharger combines the advantages of a supercharger and a turbocharger as well as eliminating some of their individual disadvantages**

supercharger and a turbocharger while eliminating some of their individual disadvantages.

The rationale for design consisted of a comparison of double stage turbocompound, single stage turbocompounding, and electrical turbocompounding to the already established single stage conventional turbocharger setup. It was decided that the thermal sensitivity and added inertia of the electrical system were larger drawbacks than the mechanical and packaging constraints inherent with the single stage mechanical SuperTurbocharger concept.

Initial modeling work was done with GT-Power on an 11 liter Hyundai natural gas engine<sup>(17)</sup>. The modeling showed an improvement in brake specific fuel consumption (BSFC) of as much as 10% for high speed and load as shown in Figure 1.18. This was due in part to the extra 22kW of power available from the turbine shaft, which was estimated to make up approximately 6% of the total power output.

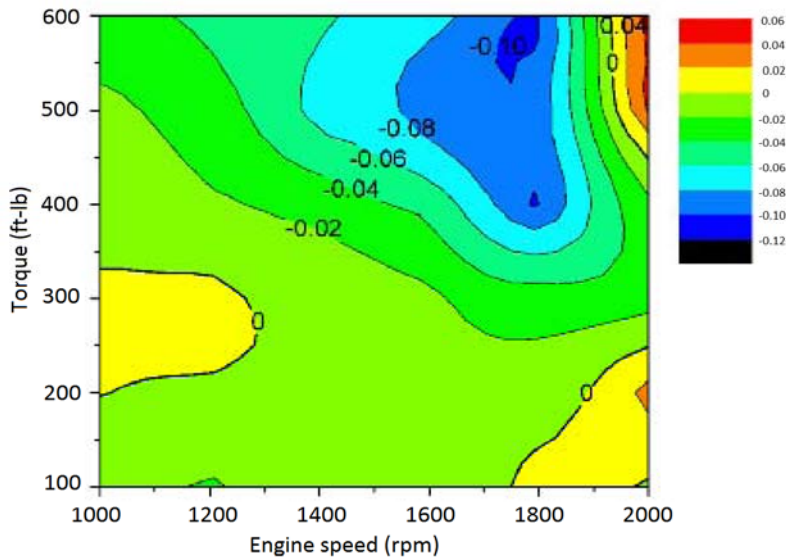


**Figure 1.19: Simulated improvement in BSFC of Hyundai natural gas engine with mechanical/hydraulic transmission to transfer power between crankshaft and turbine shaft<sup>(17)</sup>**

The Hyundai engine was not available for testing, so a Mack E7G engine with similar geometry and performance was selected. The turbocharger selected was not ideal for this engine, but was considered over others due to the variable

nozzle turbine feature that increased the controllable parameters. The turbocharger was left unmodified except for the turbo shaft, which was modified for a planetary gear arrangement. Once the speed was reduced significantly from the planetary gear arrangement, it was connected to the crankshaft through a continuously variable hydraulic transmission.

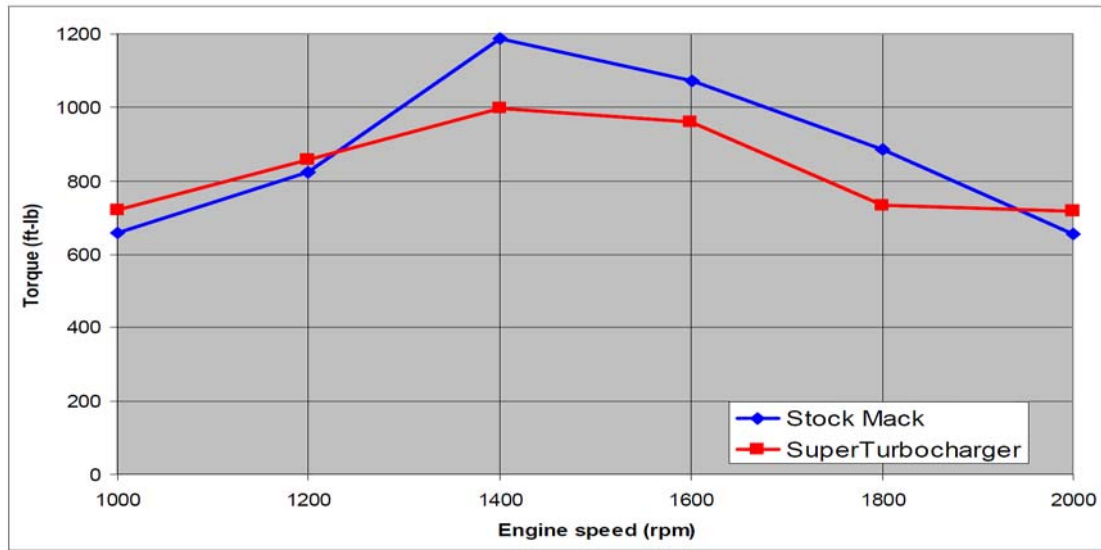
Testing showed an almost 6% gain in BSFC at high engine speed and load as shown in Figure 1.19, while also showing reduced efficiency at other operating points. These results were



**Figure 1.18: Improvement in BSFC for Mack E7G engine with SuperTurbocharger compared to stock turbocharger<sup>(17)</sup>**

consistent with the belief that the current turbocompounding setup was not as optimized as it would need to be to fully realize these benefits due to turbocharger miss-match and the use of off the shelf

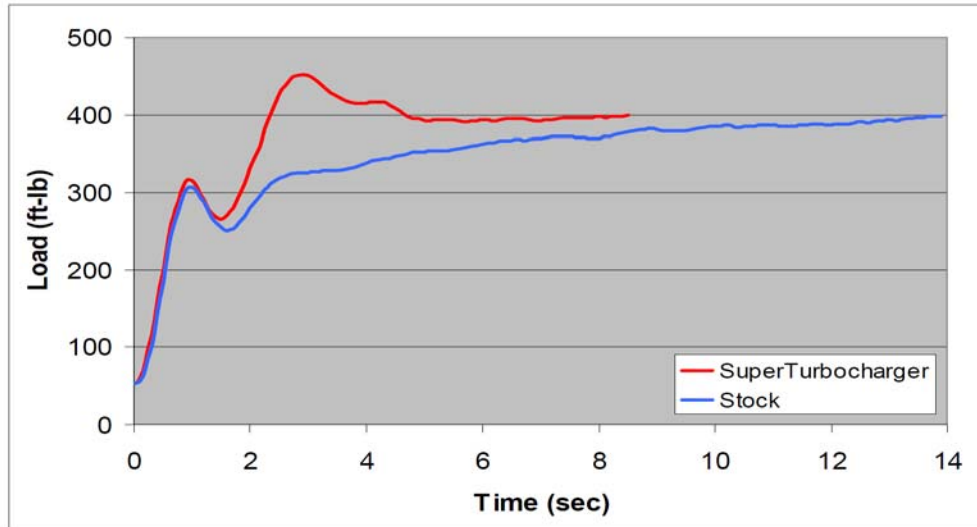
hydraulic transmission components. Wide open throttle (WOT) results shown in Figure 1.20



**Figure 1.20: WOT curve for MACK E7G with SuperTurbocharger and stock turbocharger** <sup>(17)</sup>

indicate a slight improvement in torque at lower speeds. These results were thought to be less than optimal due to the pressure limits on the hydraulic system as well as the overall limits of the speed ratio between the turbine and crank shaft. During a load acceptance test from 50 ft-lb to 400 ft-lb at 1600 RPM, the two methods were almost identical in load acceptance up to 300 ft-lb. The major difference was once boosting was required to get from 300 to 400 ft-lb, the SuperTurbo application was able to accept the load in less than 1.5 seconds due to the supercharging effect of using crankshaft power to supplement turbine power, while the regular turbocharger application took almost 13 seconds to reach full load as shown in Figure 1.21.

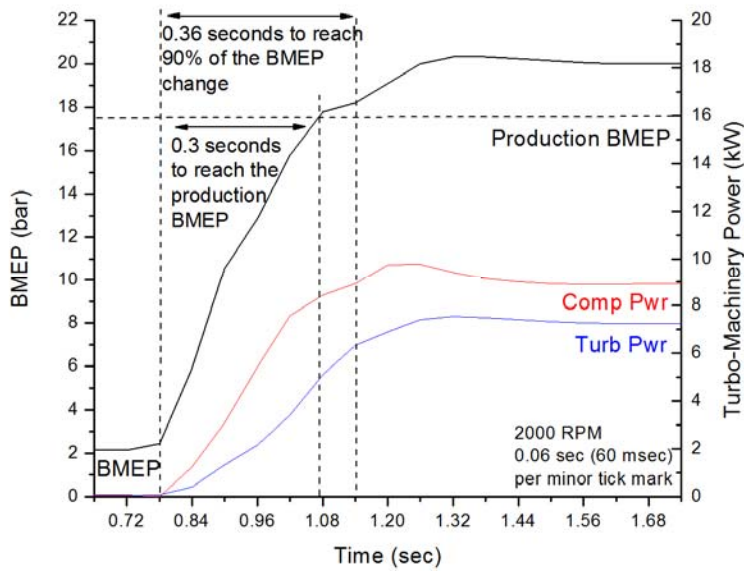
Some challenges realized were the turbocharger mismatch due to unavailability of turbo maps from the manufacturer, as well as the inefficiencies of the hydraulic transmission chosen for this application due to non-optimized operating points <sup>(17)</sup>. Since its first prototype demonstration, VanDyne SuperTurbo has spun out of its parent company Woodward Governor to focus more on automobile applications.



**Figure 1.21: Step change response of a SuperTurbocharger compared to stock turbo <sup>(17)</sup>**

Current work has focused on development of a SuperTurbo application for a 2L Volkswagen inline 4 cylinder engine as a downsized candidate for replacing a larger 3.2L V6 engine. Using engine data to compare the SuperTurbo L4 with a naturally aspirated 3.2L V6, it was estimated that the EPA city fuel efficiency would increase by as much as 17% <sup>(6)</sup>. Transient load simulation tests showed a time of 0.3 seconds to reach production Brake Mean Effective Pressure (BMEP) limit and 0.36 seconds to reach 90% of the peak BMEP for this configuration as shown in Figure 1.22. As shown in Figure 1.23, the WOT torque curve of the SuperTurbo L4 engine exceeds the naturally aspirated V6 except for the 1000 RPM operating point which is likely not to be experienced by the engine due to the stall speed limit on the torque converter and the fact that the vehicle's acceleration will be accompanied by an increase in engine speed. At wide open throttle the BSFC was higher on the SuperTurbo version as expected due to the need for retarded timing to prevent knock, which led to higher temperatures than the turbine could stand, which in turn led to fuel enrichment and higher BSFC numbers as shown in Figure 1.24. Although the BSFC numbers were higher than the naturally aspirated configuration, they

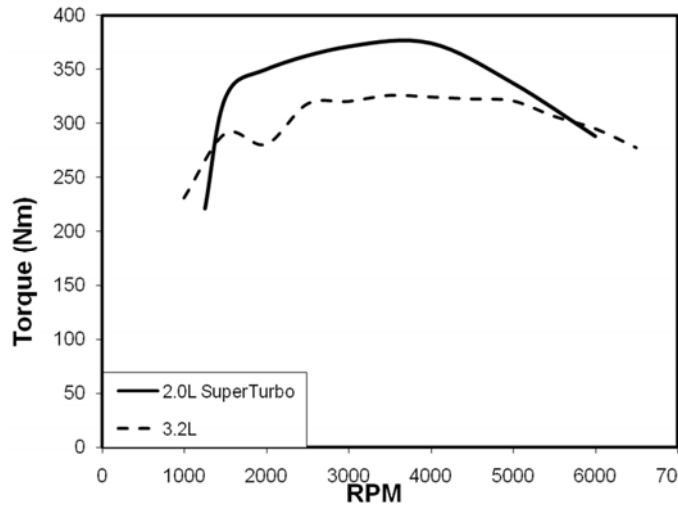




**Figure 1.23: SuperTurbo transient response for a pedal snap from 2-bar BMEP to WOT at 2000 RPM. The engine reaches 90% of the peak in 0.36 seconds<sup>(17)</sup>**

were still lower than expected stock engine values due to the added advantage of turbocompounding.

The current development of the SuperTurbo consists of a new design for the high-speed section of the transmission as well as new concepts for the continuously variable transmission

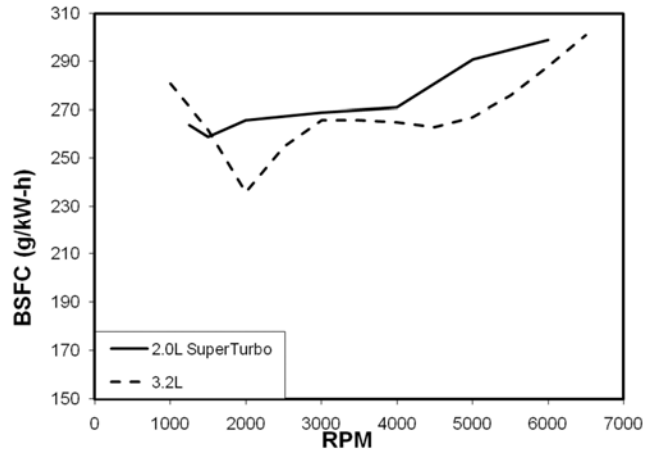


**Figure 1.22: Torque curve of the SuperTurbocharged 2.0 Liter versus a N/A 3.2 Liter V6<sup>(17)</sup>**

(CVT) part of the SuperTurbo.

The high speed section of the transmission makes use of traction fluid instead of gears for the planetary set-up which is supposed to be more efficient at the high rotational speeds associated with a turbocharger.

The focus of this



**Figure 1.24: Full load efficiency curve of the SuperTurbocharged 2.0 Liter versus a N/A 3.2 Liter V6 <sup>(17)</sup>**

project will be the calibration and use of an already developed engine model with a SuperTurbo model added in to see what efficiency gains are possible with this technology.

## CHAPTER 2: ENGINE MODEL

### 2.1 GT-Power

Engine simulation software is becoming increasingly common as evidenced by its use in the above examples. Its iterative use during the phases of engine concept, design, and development for a turbocharged engine has been thoroughly investigated<sup>(20)</sup>. Most engine simulation software makes use of 1D flow simulation to approximate the flow behavior due to the highly reduced computational demand. With simulation software, the performance of new concepts can be investigated during the design process. These models are often based on previously developed models from existing engines. Although this is an approximation, it can still provide valuable insight into concept performance at a low cost. The major computations involve the gas conditions in the pipes. This leads to a calculation of the trapped air mass used in the combustion calculation to determine the indicated mean effective pressure (IMEP). Friction and pumping losses (FMPEP and PMEP) are subtracted from IMEP to give predicted performance. This dependency makes the later calculations involving performance less accurate than the initial calculations about gas conditions.

A fully developed model relies on test data, which is typically not available during the initial concept stage of the engine. This requires assumptions to be made or data to be taken from elsewhere. The combustion model requires in-cylinder pressure data that must be estimated from similar engine data in the absence of test data. Until data is available for correlation, the accuracy will be affected according to how close the assumed data matches test data. Application to the design process includes optimization of the intake and exhaust

manifold, variable valve timing if available and turbocharger match. While going through the iterative process of choosing a proper turbocharger, it is important to work closely with the design engineers to realize all of the constraints in the design.

GT-Power can be a valuable tool for selecting a particular turbocharger based on the tradeoffs between minimizing the turbo lag for low speed torque and higher power output. The modeling of the compressor and turbocharger is accomplished with the use of compressor and turbine maps, which relate pressure ratios to mass flow for constant speed lines and efficiency contours. The simulation software assumes quasi-steady flow, meaning that it assumes steady flow relationships at instantaneous conditions. However, the unsteady flow that a turbine experiences is most likely outside the turbine map provided by the manufacturer. An extrapolation method is suggested by the software manufacture to account for this. When modeling the effect of a turbocharger, it is suggested that the inlet conditions be set to the compressor outlet conditions and a nozzle added to the exhaust to simulate the backpressure from a turbine. This gives a good estimate, but does not take into account the link between intake and exhaust.

Once a prototype engine is available, it is likely that test data is available to validate and update the current model to provide greater accuracy. For turbocharged engines correlation data is very important because the model is especially dependent on turbocharger performance, particularly at low speeds where the wastegate is shut <sup>(20)</sup>. As shown in Figures 2.1-2.5, a close match to test data can be modeled to within about 5% accuracy for most of the speed range of this WOT test.

Discrepancies in volumetric efficiency are attributed to the combustion model as shown in Figure 2.1. The difference between compressor pressure outputs in Figure 2.4 is due to the difference in tested and manufacturer supplied compressor efficiency. The heat transfer model

used for the exhaust is cited as the cause for turbine inlet temperature discrepancies in Figure 2.5. These discrepancies are thought to cause the air flow and torque discrepancies of Figure 2.2 and Figure 2.3. With the proper correlation data, an engine's performance can be modeled with

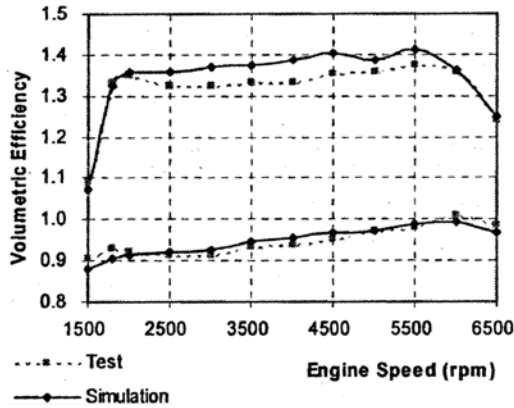


Figure 29: Volumetric efficiency correlation <sup>(20)</sup>

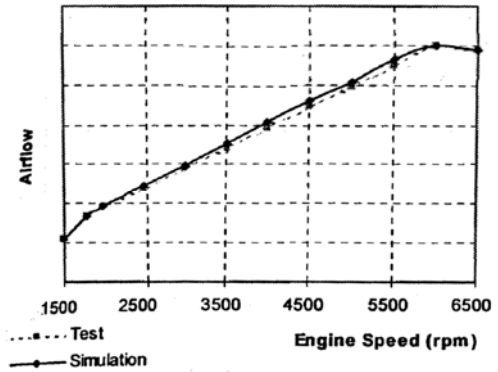


Figure 28: Airflow correlation <sup>(20)</sup>

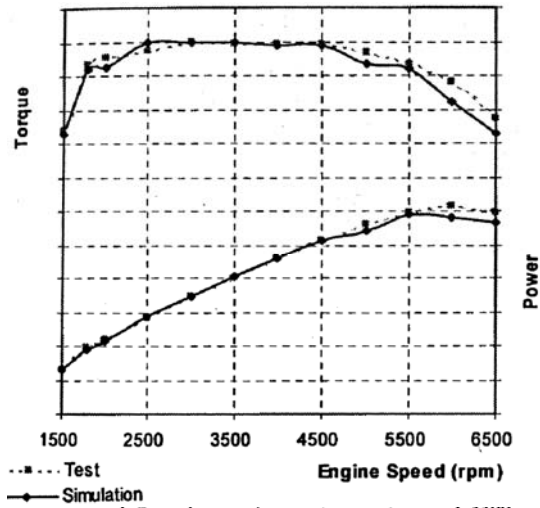


Figure 27: Performance Correlation <sup>(20)</sup>

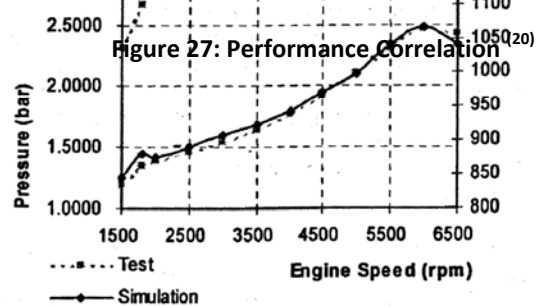


Figure 25: Turbine Inlet Conditions <sup>(20)</sup>

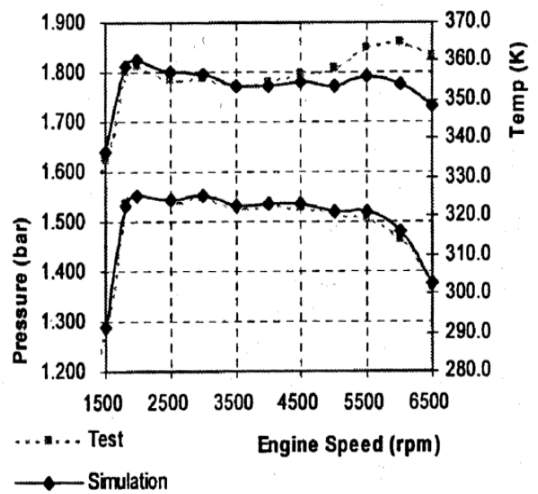


Figure 26: Compressor Outlet Conditions <sup>(20)</sup>

validated accuracy. With the proper test data to validate a model, it is still possible to investigate different design strategies that deviate from the original model, but at a reduced level of accuracy.

## 2.2 Stock model from Volkswagen

GT-Power is chosen as the modeling software due to the availability of a previously developed stock engine model from Volkswagen. The original model Southwest Research Institute (SwRI) received of a 2.0L L4 turbocharged GDI engine is validated against test data <sup>(6)</sup>. Although it is cited as being validated, it is unknown to what extent the original model is validated. Additional validation is carried out for this project using a more extensive data set provided by Volkswagen that covers the entire speed and load range. The original model did not have a throttle so it is likely only validated against WOT data with the timing parameters set for WOT operation only. The data provided by Volkswagen is contained in Appendix A. A map view of the simulation layout along with a map view of the controls layout is shown in Figures 2.6 and

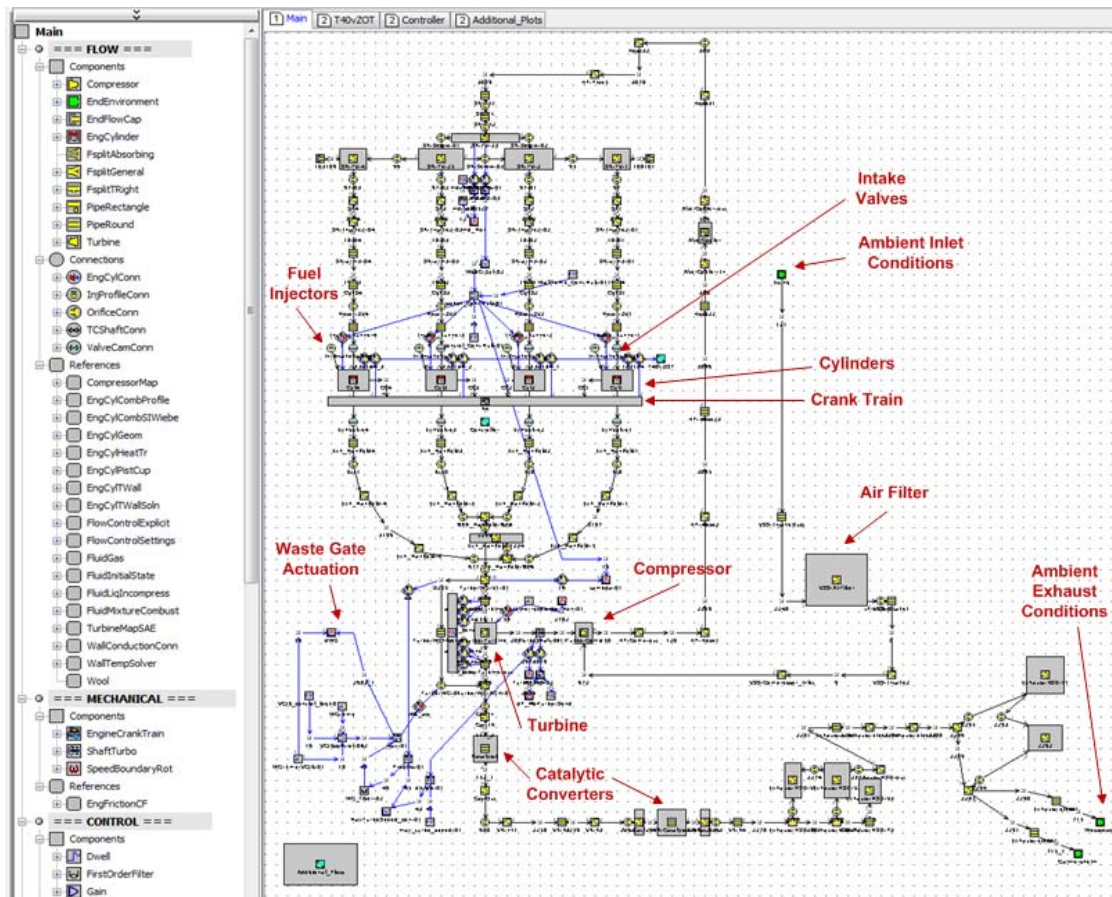
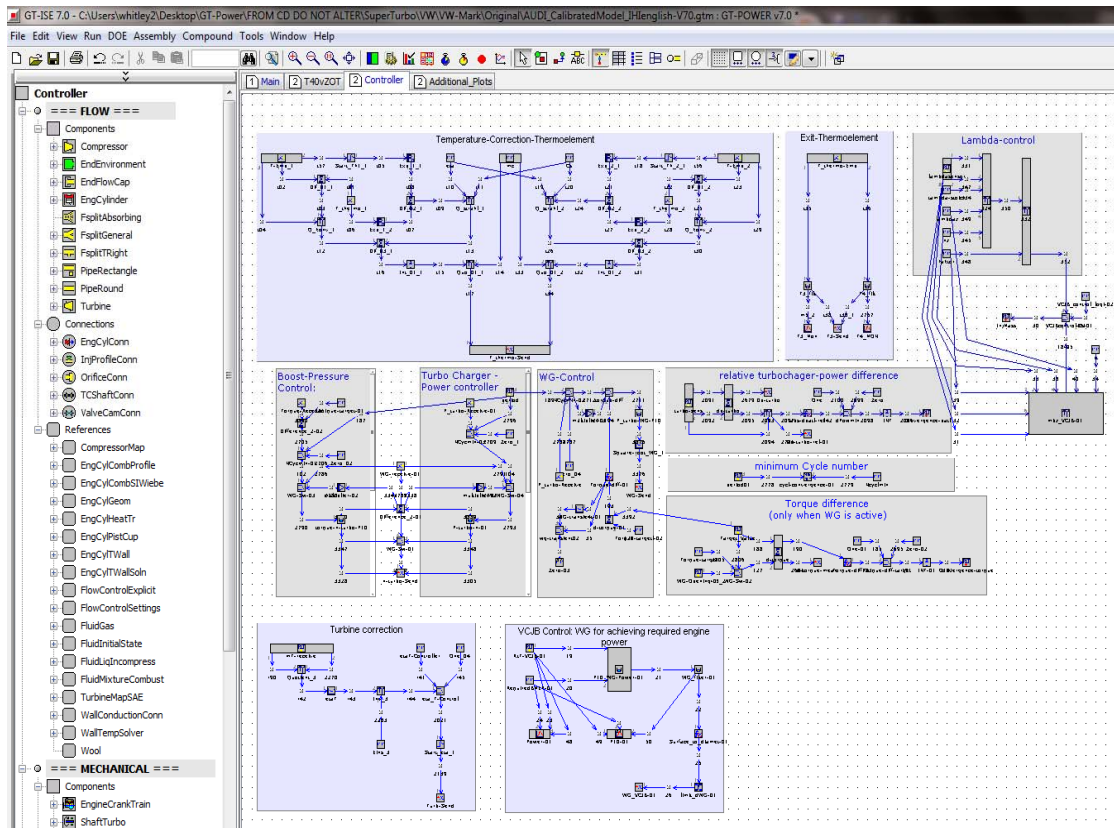


Figure 30: Main View of GT-Power Model in original form from Volkswagen

2.7, respectively. The map view of the engine model has several of the major components labeled. On the left side of the map view, the tree view shows all of the parts currently set up to use in the model. The majority of the components used are pipe objects as they represent the flow boundaries throughout the system.



**Figure 31: View of the controls page dealing with temperature corrections, wastegate control, and lambda control.**

For this original model the wastegate is set up to control desired engine brake mean effective pressure (BMEP) since there is no throttle. For this project a throttle is added and the wastegate is used to control boost pressure rather than BMEP so that the throttle is able to independently control BMEP. The original model contains two sets of maps for both the turbine and compressor.

The GT-POWER model is set up to run until it converges based on several requirements specified in the Run Setup menu. There are a maximum number of periods or engine cycles the

user must specify that the simulation will not go over, but it will not necessarily converge by this point. The output file provides information on the number of cycles run for each case and convergence. There are also a minimum number of cycles the user can set that the simulation must run before being allowed to stop even if all the convergence parameters are met. In the flow convergence tab a mass flow steady state auto shut off limit is set. There is another tab that lets the user select which result (RLT) variable should converge before the simulation ends. RLT variables are the single valued results such as averages, maximums, and minimums during a cycle as opposed to instantaneous values which are usually plotted versus crank angle or time. The RLT variables that are listed to define convergence are the turbine shaft speed, the shaft torque imbalance between the compressor and the turbine, BMEP, and the orifice diameter of the wastegate. With the exception of the shaft torque imbalance being set to target zero, the parameters are set to converge on a cycle-to-cycle basis all within a certain user defined tolerance.

### **2.3 SuperTurbo added by SwRI**

The main addition to the model by SwRI is the transmission between the turbocharger and the engine crankshaft. In some of SwRI models the wastegate is removed while it remains in other versions of the model from SwRI. The most used version has a gear ratio controller that would adjust the gear ratio according to a desired air mass flow rate through the compressor. The addition of the CVT to the SuperTurbo transmission presents challenges with torque oscillations due to the cyclic behavior of the engine and the pressure pulses on the turbine. It is assumed during the initial modeling done by SwRI that the final version of the transmission will have some way of dampening these oscillations <sup>(6)</sup>. This method is not yet finalized so the SwRI model without the gear ratio control and with wastegate control is used for this project. To deal



with the torque oscillations the SwRI model senses a torque value from the SuperTurbo shaft. These sensed instantaneous torque value are then averaged over two cycles at an interval of twelve times per cycle. This averaged value is multiplied by the efficiency if it is positive and divided by the efficiency if it is negative. This final value is imposed on the shaft labeled ST\_TQeff\_shaft\_01 with a torque object which is in turn connected to the CVT transmission as shown in Figure 2.8. In addition to the assumption that the final version of the CVT will include

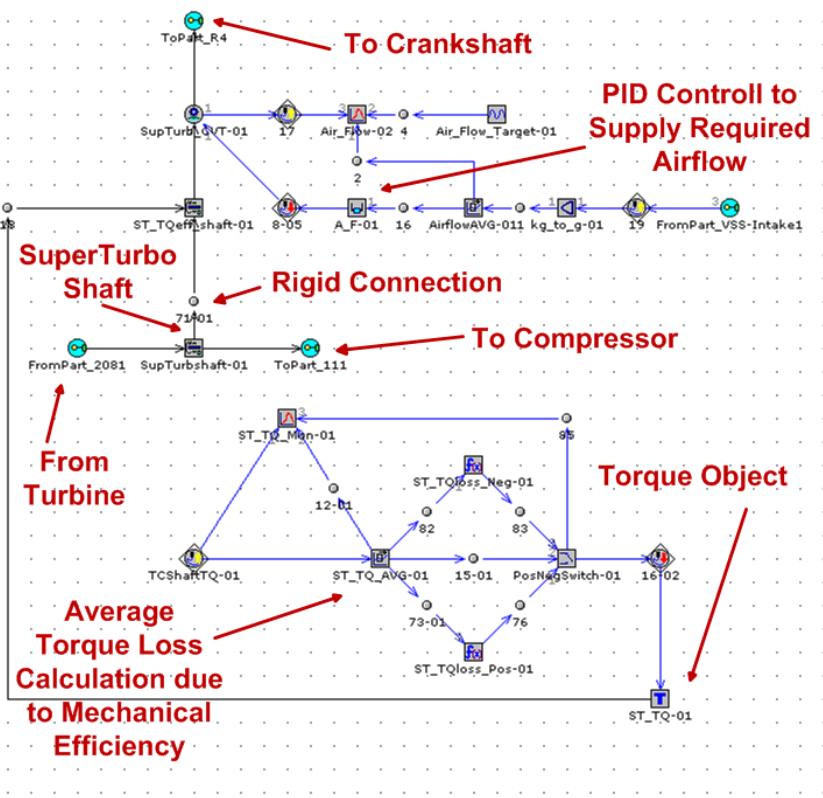


Figure 32: SuperTurbo Model added by SwRI

damping, it is also assumed that the mechanical efficiency will be 80%. GT-Power defines the efficiency based on torque so that torque out will be 80% of the torque in value. This efficiency is an estimation and

will need to be modified once data is available on efficiency versus speed, load, gear ratio and parasitic losses.

## 2.4 Stock Engine Model Tuning

In addition to the modification to the SuperTurbo model, there are several changes that need to be made in order to compare the stock simulation against the stock test data. The original model is set to operate at wide open throttle, so it has no information or controls to allow operation at part load or throttled operation. In addition, the engine has several parameters that change depending on both load and engine speed such as ignition timing, camshaft timing, and injection timing. An example of these timing events relative to crank angle and piston location is shown in Figure 2.9. This extended validation concentrates on getting a

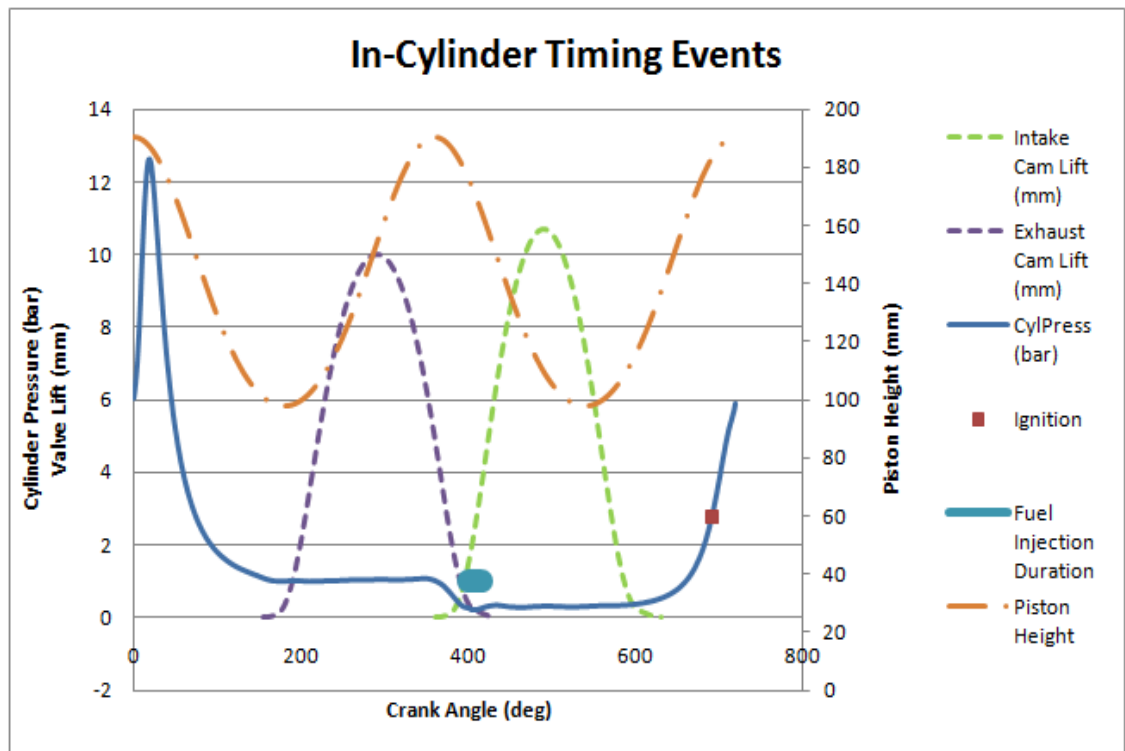


Figure 33: In-Cylinder Timing Events

close match for BMEP and BSFC while also checking that other parameters do not have excessive discrepancies.

In order to isolate different tuning effects the original model is stripped down to start at the intake manifold and end at the exhaust manifold. This version of the model is labeled m1.

Intake and exhaust manifold pressures and temperatures are available from test data to set as ambient intake and exhaust conditions for this configuration. This configuration allows for isolated tuning of parameters within this flow path of the engine.

Another advantage model m1 has over the full engine model is the elimination of the turbocharger and associated wastegate control. The wastegate control and turbo lag associated with the turbocharger's large inertia take a considerable amount of computational time to converge to steady state values. The elimination of this part of the model allows for much faster tuning of the parameters that are independent of the turbocharger operation except for minor pressure pulse effects.

To get the most accurate comparison several parameters from the test data are specified for the engine in the design of experiment (DOE) setup folder. The parameters imposed on the model from test data include compressor outlet temperature, compressor outlet pressure, turbine inlet temperature, turbine inlet pressure, ignition timing, fuel injection timing, fuel injection duration, lambda, camshaft timing, and engine speed. Since the original model is set up for WOT operation only, it has no information or controls for these parameters during part load operation.

The lambda values available from test data are imposed on the model as targets. In the controls page of the model there is a group of objects used to control the amount of fuel injected based on the desired lambda value and the air flow through the engine. For turbocharged engines this is an especially important aspect since fuel is often used to cool down the turbine at high engine speeds and loads. If the temperature limit is exceeded the turbine can expand and cause catastrophic failure. Extra fueling is also sometimes used to prevent pre-detonation. The injected fuel mass is an important parameter for characterizing the improvements in BSFC.

The first parameter to tune in model version m1 is the camshaft timing data. While all of the other timing parameters are referenced relative to the crank shaft top dead center (TDC) position the crankshaft timing data is not referenced to anything. The timing has to be referenced to before or after TDC and can be referenced to the crankshaft angle or the camshaft angle. All of these possibilities are run and the case with the closest BMEP match to the test data is chosen.

While this initial tuning is able to show an overall BMEP trend match to test data for the camshaft timing reference chosen, the model is not able to reach as high of values for BMEP as the test engine is able to at lower loads as depicted in Figure 2.10. The error is larger as the

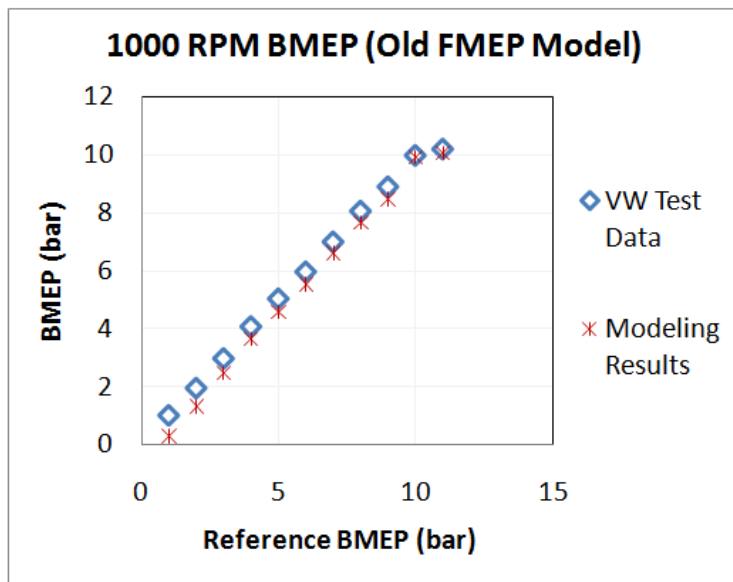


Figure 34: Error in BMEP values for modeling data compared to test data

reference BMEP is reduced. Analysis of IMEP, trapped air mass and other parameters indicate that the FMEP model is inaccurate. At higher loads this difference is not as significant since FMEP is a smaller fraction of IMEP. GT-Power uses the Chen-

Flynn engine friction model to calculate FMEP values as shown below

$$\text{FMEP (bar)} = C + \text{PCPF} + \text{MPSF} + \text{MPSSF}$$

where,

C = Constant Term

PCPF = Peak Cylinder Pressure Factor

MPSF = Mean Piston Speed Factor

MPSSF = Mean Piston Speed Squared Factor

The C value for the original model is set at 0.8443 bar when the recommended value for this is suggested by the GT-Power Help Navigator to be between 0.3 and 0.5 bar. Several different versions of FMEP were modeled by varying the different coefficients to be mostly within the range suggested by GT-Power in an effort to lower the error in the low BMEP range of the map. The old and new output of the FMEP model is shown in Figure 2.11. Ideally there

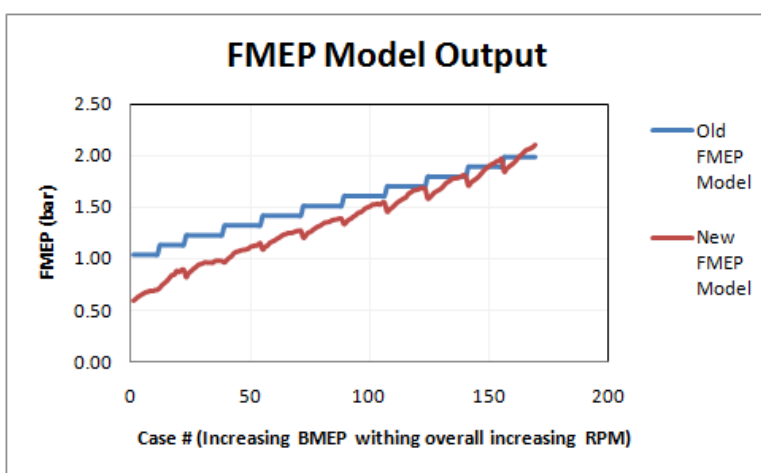
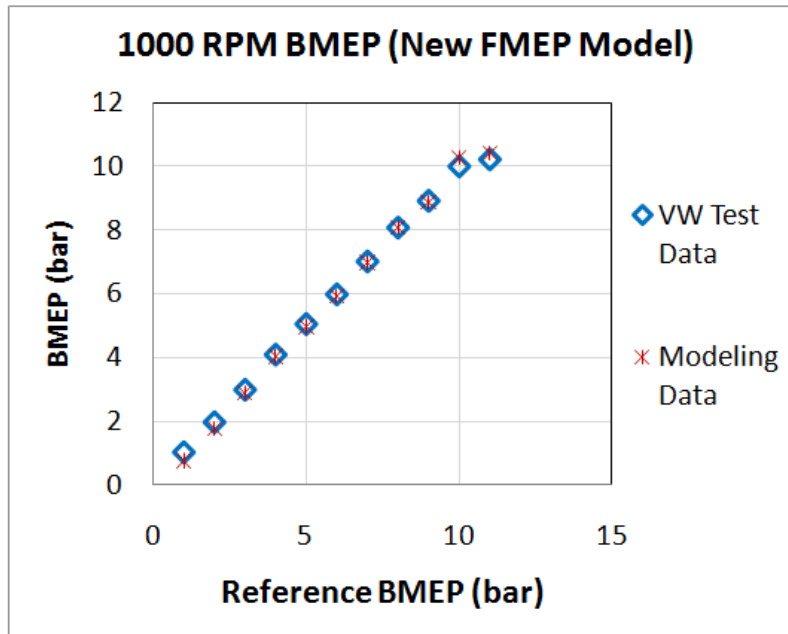


Figure 35: FMEP Model Output from GT-Power

acceptable levels as shown in Figure 2.12.

The next step in tuning the stock model consists of adding a throttle component to the model and extending the inlet conditions back to the outlet of the compressor. The addition of the throttle makes this model m2. The new inlet conditions are then set to the outlet conditions of the compressor based on the available test data. The flow through the throttle is based on an array of discharge coefficients versus throttle angles. The discharge coefficients are defined by the ratio of effective flow area to the reference flow area which originates from the isentropic velocity equation for flow through orifices. The flow coefficients were determined as follows: First, an initial guess is used and a simulation ran over the BMEP range for an engine speed of

would be test data for FMEP values so that the FMEP model could be evaluated directly, but unfortunately this data is not available. After this change the BMEP error comes down to more



**Figure 36: Error in BMEP values for modeling data compared to test data**

1000 RPM at the throttle angles listed in the test data. The model predicted pressure drop values were compared with test data and then the flow coefficients were adjusted iteratively until a close match is achieved. This is done

for 1500 RPM as well and then the results were applied to the whole speed range to verify that they are a good match for the entire speed range. GT-Power has an Excel spreadsheet that calculates coefficients automatically if the specified test data is available.

The final step of validating the model against test data requires the addition of the turbocharger into the model (model m3 and m4). As previously mentioned the wastegate in the original model is used to control BMEP instead of the more typical configuration of controlling boost pressure. Since the throttle is controlling BMEP, the wastegate controller is altered to control the desired boost pressure sensed after the intercooler. The reference boost pressure is based on post intercooler pressure measurements from the test data.

The final comparison will consist of two configurations. In the first round of this final comparison the throttle angle will continue to be specified based on the test data (model m3) while the second round will use a throttle controller object that will adjust to give the desired BMEP (model m4). This will allow for the comparison of matching parameters based on the

exact throttle angle, and a slightly different version where the throttle angle can vary to give the referenced BMEP values.

The test data imposed on this simulation configuration include ignition timing, injection timing, injection duration, camshaft timing, lambda values, and engine speed as well as the throttle angle added for the previous configuration. In addition to these, reference boost pressure sensed after the intercooler is added as a target for the wastegate controller.

Depending on the simulation type mentioned above, reference BMEP will also be used in place of the imposed throttle angles and the throttle will be controlled to try to match this reference BMEP value.

The results from these simulations show a close overall match to the test data for both configuration setups. While the first version has larger BMEP difference, the second one has throttle angle differences as expected. The for the configuration where throttle angle was adjusted to match BMEP values the average BSFC error is about 3.2% with a standard deviation of 4.3%.

Even though most of the parameters of importance match well with test data, the exhaust temperatures are much higher than those reported in the test data. This would have been apparent with the earlier models, except this parameter is not directly compared upstream of the boundary conditions that are specified to match the test data in model m1. It appears that there are heat transfer discrepancies between the model and the test data that have not been accounted for since the individual exhaust runners were all indicating higher exhaust temperature directly out of the cylinder model. This difference is likely due to several issues unrelated to the heat transfer model of the cylinder. The precise location of the measurement is unknown. The measurements themselves are difficult to interpret due to radiation and conduction losses that need to be accounted for. Additionally, the temperature at

this location is undergoing rapid oscillations. The errors in the GT-Power model as they correlate to the given test data are also dependant on the inherent error in the actual measuring of the test data. GT-Power's manual indicates that this is to be expected and correlations are given to how much these exhaust temperatures are likely to be off depending on different wall and ambient air temperatures.

Another noticeable difference is the modeling output of the turbine pressure and temperature drops across the turbine. More research into the details of the GT-Power turbine model reveals some possible reasons for these differences. The turbine map data entered into GT-Power goes through pre-processing which extrapolates the data over a wider range and fits it to a curve for use in the actual simulation. In post-processing there are plots which indicate how well the turbine data is suited for this processing. Analysis of the plots reveals that both sets of turbine data provided in the original model are for a very narrow range compared to the range that the turbine data will be extrapolated out to as shown in Figure 2.13. Data for the first turbine map does not fit the normalized curve as well as the second turbine map and the pressure ratio curve for the second turbine map also gives a better fit as shown in Figure 2.14. The efficiency curve of the second map is less smooth and shows two local maximums when it is desirable to have only one maximum.

This leads to an experiment comparing two simulation runs with the two different turbine maps. The results from the second turbine map are very similar to the results of the first map with a few exceptions. The second turbine map is unable to produce results for a few of the low load, low speed cases. This is presumably due to the fact that the data seems to be slightly more towards the higher end of the blade speed ratio (BSR) line causing the low speed, low load operating points to use data extrapolated out to a distance that is unreliable. The second map does not match the temperature



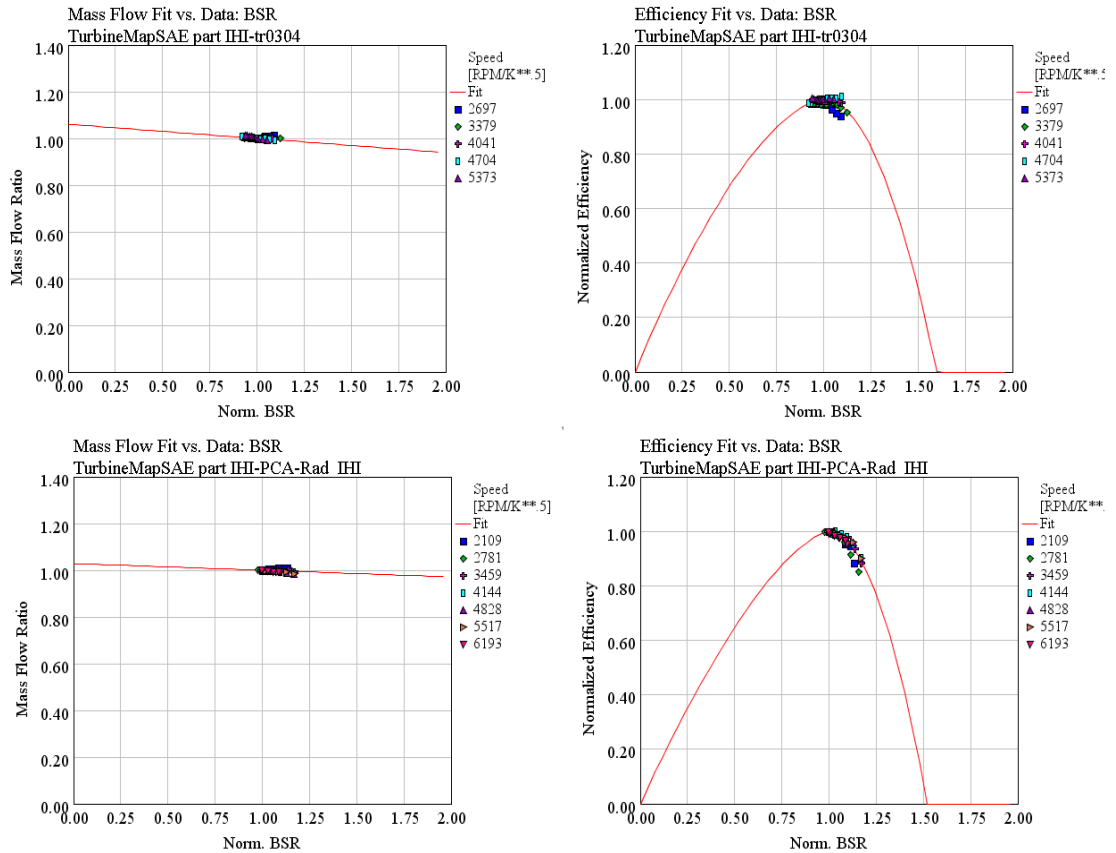


Figure 37: Turbine Map Data Analysis Plots for Turbine Map 1 (above) and Turbine Map 2 (below)

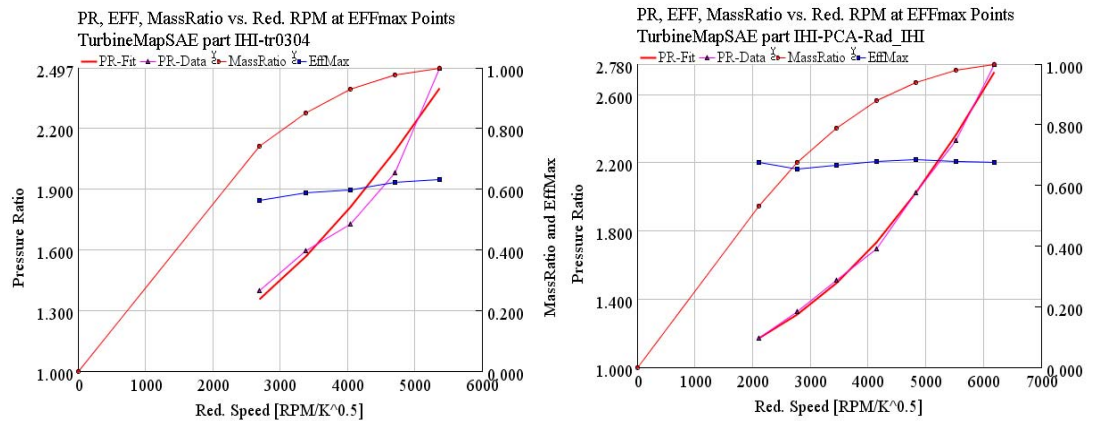


Figure 38: Turbine Map Data Analysis Plots for Turbine Map 1 (left) and Turbine Map 2 (right); the Blade Speed Ratio (BSR) is used for normalization and extrapolation calculations in GT-Power and pressure drops across the turbine as well as the first turbine map, although the difference between the two are much smaller than the overall difference from the test data. Since the second map does not match as well the best option is to continue using the first map. Despite

these obvious faults of the model, they will still serve as a useful tool for evaluating a comparison between the original stock engine model and a SuperTurbo version of the model.

While it is preferable to have a more extensive set of data that will allow for more precise correlation, as well as more information about the iterative build-up of the model, the current model will still be adequate in serving its purpose of showing improvements to the SuperTurbo model over the stock engine model.

## **CHAPTER 3: ENGINE SIMULATIONS**

With a stock simulation model ready for use simulations are run to see where efficiency improvements can be made and then the SuperTurbo version is ran for comparison. The stock model is modified to be able to run at engine speed and load points not listed in the test data. Stock model runs at these new data points are analyzed to see what areas of the BSFC map may offer the most improvement. The SuperTurbo is added to the model to evaluate the improvements.

### **3.1 Simulation of Stock Engine Configuration**

For the remaining simulations the engine model is modified to be able to run at operating points other than those listed in the test data. This allows operation between test data points where the exact load and speed dependant parameters are not provided. The new data points are at BMEP intervals of one bar and engine speed intervals of 500 RPM. The results from this simulation are analyzed to see areas of potential BSFC improvements.

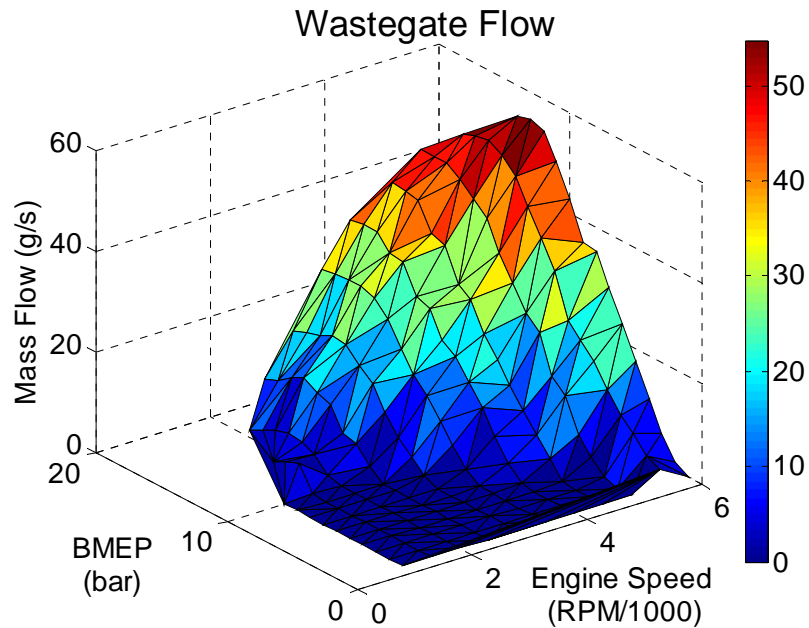
Modifying the model to be able to run at points other than those listed in the test data is accomplished through the use of the neural network tool in GT-Power. Neural networks consist of interconnected processing elements called neurons which work together in parallel to compute an output based off one or more inputs. Neural networks are trained to interpolate the nonlinear behavior of important engine parameters that vary based on engine load and speed. The neural network tool consists of a training tool and a neural network control component for use in the simulation model. The neural network training tool teaches a neural

network the desired output based on a given set of inputs. The use of neural networks instead of a simple look-up table has several advantages such as being able to determine the output in a shorter amount of time, being able to handle data where linear interpolation is insufficient, and being able to handle more inputs.

The different training methods are briefly explained in the Train Neural Network document provided with the software <sup>(21)</sup>. Once the neural network is trained, a file is created that can be accessed by the neural network control component for use in any model. To train the neural network, the user specifies input data such as BMEP and engine speed values. The desired output values are entered, such as camshaft timing, ignition timing, etc. The user selects one of the various training methods or lets the software automatically choose one. If automatic is chosen the software runs several training methods and output the results of the method that provided the lowest root mean square (RMS) training and testing error. As a built in default, the training methods set aside 10% of the data to validate the training once the training is complete. Parameter modeling using this method consists of ignition timing, injection timing, injection duration, camshaft timing, lambda values, reference boost pressure, and reference BMEP.

With a more general model capable of running at any point within the range of engine speeds and BMEPs, the model is run for comparison against a SuperTurbo configuration. Since wastegate data is not included in the test data, the modeling is split into two sections with overlap in the transition area. One section contains data points where the wastegate is closed. The second set uses the wastegate to control desired boost.

Based on results from the stock engine modeling there are several areas of possible BSFC improvement. The first case occurs when the wastegate is open. With typical turbocharger configurations the wastegate is open when the exhaust gas supplies the turbine with more energy than the compressor is able to effectively use. This area of the map covers the higher

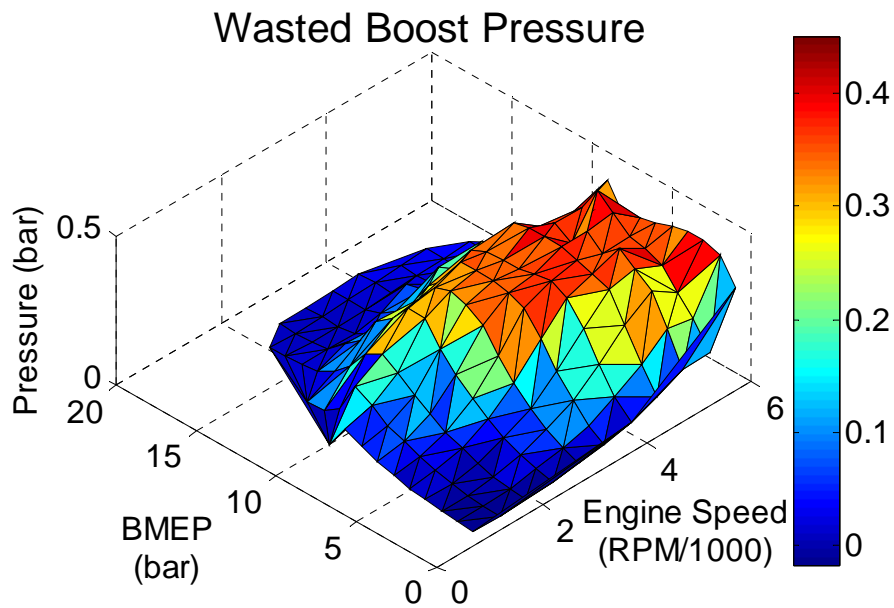


**Figure 39: Wastegate flow versus engine speed (RPM/1000) and load (bar)**

engine speed and load regions of the engines operating range as shown in Figure 3.1. The wasted exhaust gas flow corresponds to the additional energy available that can be used for turbocompounding with the red highest flow area able to show the most improvement. The wastegate diameter is evaluated using the model by controlling the diameter based on desired boost pressure input from the test data available. With the SuperTurbo configuration the wastegate is closed and the gear ratio is controlled to act as a brake on the turbine so that it does not over-speed. This braking energy is fed back to the crankshaft through the transmission.

The second case occurs when the compressor is creating boost pressure that is not needed. The compressor uses energy from the turbine to create boost and the throttle is actuated to control engine load so that much of the energy spent on creating boost is lost through the pressure drop across the throttle. This is done with the stock configuration to keep turbine speeds high. Even though boost is not needed the stock configuration creates the boost in anticipation that boost will be needed soon. The turbo speed is kept high and a recirculation valve is used when the boost is not needed. Once boost is needed the turbo is already at or near

a speed where it is able to adequately provide boost so that turbo lag is less of an issue. The area of the map with boost pressure higher than atmospheric pressure and manifold pressure less than atmospheric pressure covers a large area of the map except for high load cases where the throttle is at or near wide open position and low load and speed cases where the boost pressure is already near atmospheric as shown in Figure 3.2. The wasted boost pressure is calculated by subtracting manifold pressure from boost pressure if manifold pressure is greater



**Figure 40: Boost pressure created by the compressor that is unneeded** than atmospheric pressure and by subtracting atmospheric pressure from boost pressure when manifold pressure is lower than atmospheric pressure. The compressor is wasting the most power in the red higher pressure difference area so it should show the most improvement.

When boost pressure is not needed the compressor can be controlled to provide almost no boost with the braking energy going to the crankshaft instead of being wasted. Reduced low load compressor use also has the advantage of supplying the cylinder with cooler air. Since the air is not being compressed as much before passing through the throttle restriction the intake air temperature is denser due to its lower temperature.

Another source of improvement is the substitution of the stock turbine design for a new turbine design that is optimized for this type of application. Turbochargers have a tradeoff between turbine efficiency and turbo lag. The turbine must be larger in order to be more efficient causing the turbo lag to increase. With the SuperTurbo design the turbine inertia is less of an issue since engine power is available to accelerate the turbine so the size can be increased to make it more efficient.

### **3.2 Comparison of SuperTurbo Model to SuperTurbo Test Data**

While development of the SuperTurbo is in the early stages some test data is available which demonstrates turbocompounding ability. The configuration consists of the high speed section of the SuperTurbo and a fixed gear ratio pulley between the high-speed section and the crankshaft. In addition it utilizes the lower efficiency stock turbine. The test data currently available uses a pulley from a 4000 RPM max load test at 2707 RPM part load conditions. Unfortunately, the engine speed and load tested do not deviate much in ratio from the 4000 RPM full load condition so the improvement is not as much as it could be for this operating point. While there is some turbocompounding benefit due to the wastegate being closed the gear ratio is not optimal for this condition since the boost pressure is higher than the required manifold pressure and the throttle is not wide open. The more optimal comparison for this operating point would be at a condition where the throttle is wide open or a gear ratio where wasted boost is not created.

The model is run with the parameters available from test data imposed into the model. These include lambda, ignition timing, gear ratio, and atmospheric conditions. While not all of the results match the test data parameters that are available they are close enough of a match to provide validation in terms of turbocompounding ability. While the ignition timing is imposed

on the model from the test data, there seems to be some discrepancy with regards to the combustion model with the 50% burned crank angle location being a couple of degrees off. The BMEP and IMEP(360) data points show less than one percent difference and the IMEP(720) data shows about two and a half percent difference. However, the FMEP data show a difference of sixty percent. This is due to the way it is calculated. The BMEP value is calculated based on the brake torque measured. This gives an unrealistic number for the actual in-cylinder BMEP value since it now incorporates turbocompounding power. Since the BMEP now incorporates turbocompounding power, the calculation of FMEP based on BMEP, IMEP(360) and PMEP is off.

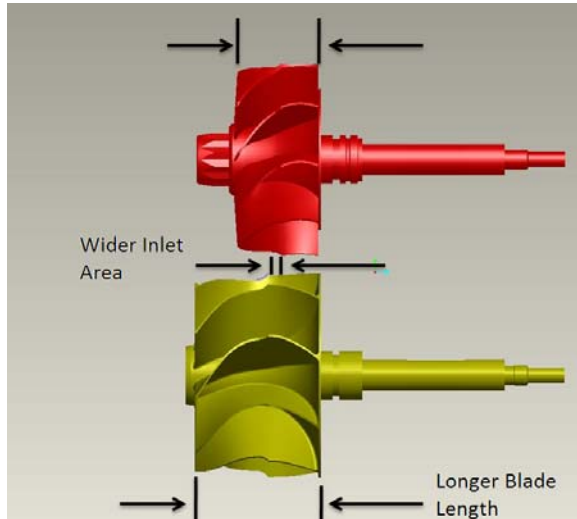
The modeling data is available for comparison in Appendix D. The BSFC values are between one and two percent off but it is still possible to determine the benefits of the turbocompounding by comparing the SuperTurbo model to the stock model at these same conditions. The BSFC improvement of the SuperTurbo model over stock model is about 1.2%. The amount of torque added to the crankshaft in the test data is 4.31 Newton meters (N-m) which is close to the modeling prediction of 4.51 N-m. This close match of transferred torque provides validation that the model is capable of predicting the improvements due to turbocompounding.

### **3.3 Simulation of SuperTurbo Engine Configuration**

The SuperTurbo configuration incorporates a new turbine wheel in production form so a different model is used for the SuperTurbo turbine. Since the design is not fully developed and tested a simplified version of a turbine is modeled. GT-Power has a simplified turbine model that predicts output power, mass flow rate and outlet temperature using an orifice flow model and imposed efficiency. The new turbine design is capable of much higher efficiencies than a normal turbocharger turbine since there is no longer a tradeoff between turbine inertia and turbo lag.



The differences in the two designs are illustrated in Figure 3.3. The figure shows that the new turbine design has noticeably longer blades and a larger inlet area. The orifice diameter in the model is determined based on matching the pre-turbine pressure of the stock configuration at



**Figure 41: The new turbine design has a wider inlet area to allow less back pressure as well as longer blades to improve efficiency**

5000 RPM full load with the SuperTurbo version at the same load, but with the wastegate closed.

The final configuration is split into a high load section where boost is needed, and a low load section where boost is not needed. The high load cases will have a wide open throttle and will use the SuperTurbo gear ratio to control engine load. The limit on how low of a load can

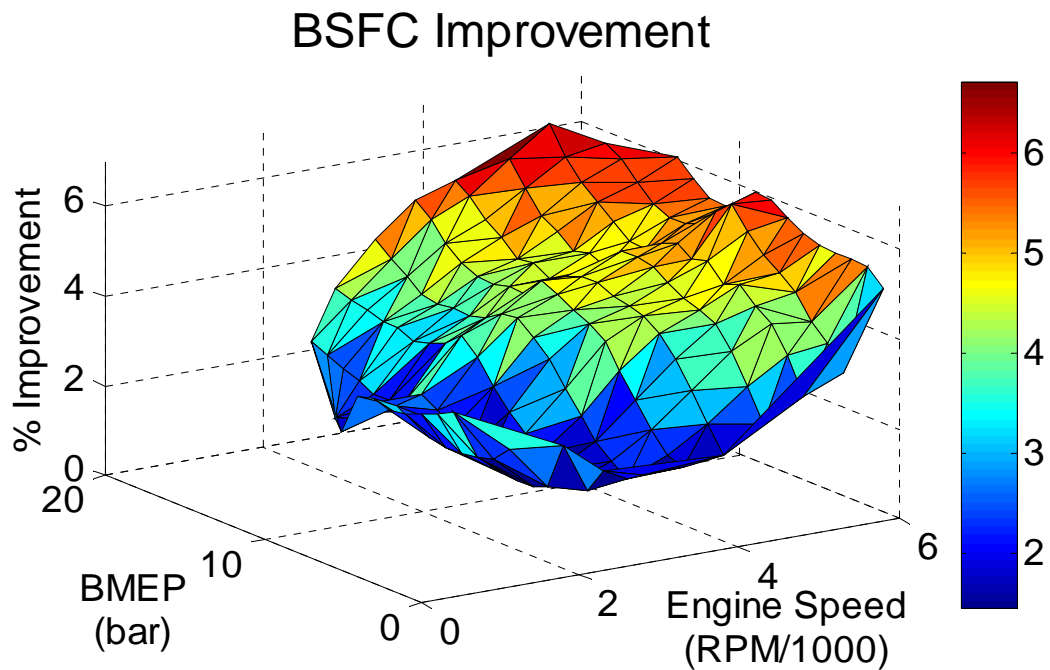
be achieved with this method is dependent on the compressor pressure ratio. Modeling of pressure ratios below one without test data is not recommended by GT-Power due to the need to extrapolate the available data out to these points. The second section uses the throttle to control load while the SuperTurbo gear ratio is used to control the speed of the compressor such that its pressure ratio remains slightly above one.

Initial modeling with the simple turbine reveals that flow convergence parameters are not converging for some cases. The flow convergence is specified in terms of change in mass and pressure flow from one period to the next at any flow path location. The location of the changing flow is further down the exhaust stream in the muffler section of the model. The source of this is thought to be due to the simple turbine having less of a pressure drop across the turbine part of the system. While the pressure drop across the turbine is the same, the stock

model's wastegate is open allowing for more of an overall pressure drop which dampens the pressure oscillations coming from the engine. Due to this the flow convergence parameters were slightly relaxed from a fractional value of .002 to .02.

### 3.4 Results

The primary parameter of interest for this modeling is brake specific fuel consumption (BSFC). The results show improvement in BSFC over the entire range of the map as shown in Figure 3.4. The improvements correspond to the predicted areas of improvement in the range of



**Figure 42: Brake specific fuel consumption improvement over stock engine**

2 – 6 % over stock conditions. The high load and speed areas are affected more by turbocompounding from the wastegate being closed as shown in Figure 3.5. The middle to lower region is affected most by the reduced losses associated with using energy to compress air that is later restricted by the throttle as Figure 3.6 illustrates.

The power transfer from the turbine to the crankshaft is higher with increasing speed and load as shown in Figure 3.7 maxing out at 6000 RPM 14 bar BMEP. This correlates well with

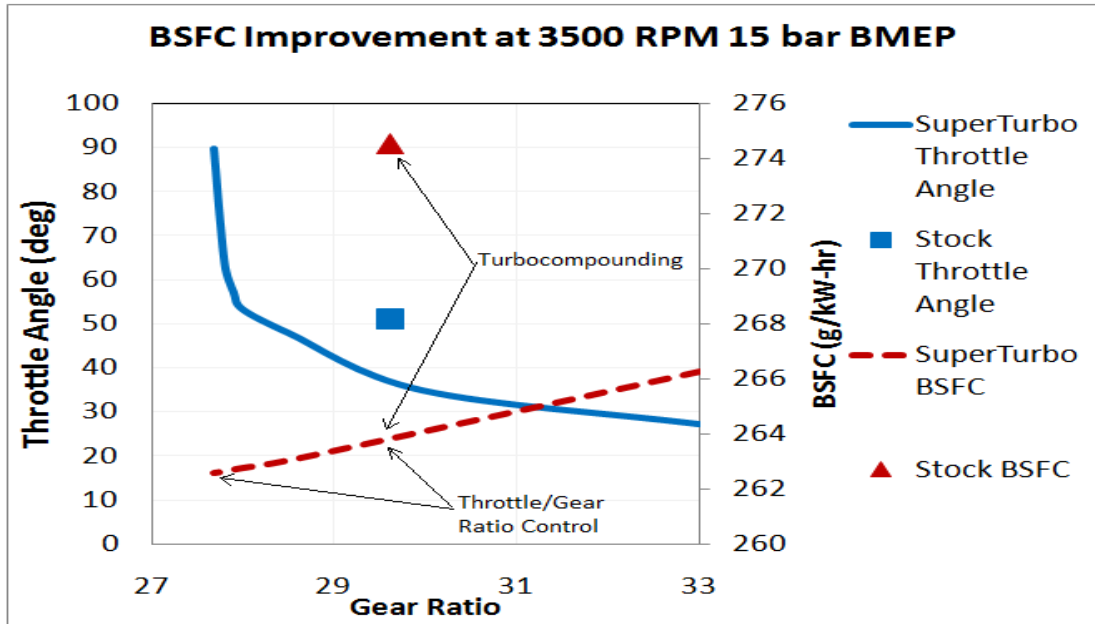


Figure 44: At this higher load point the contribution from turbocompounding dominates the

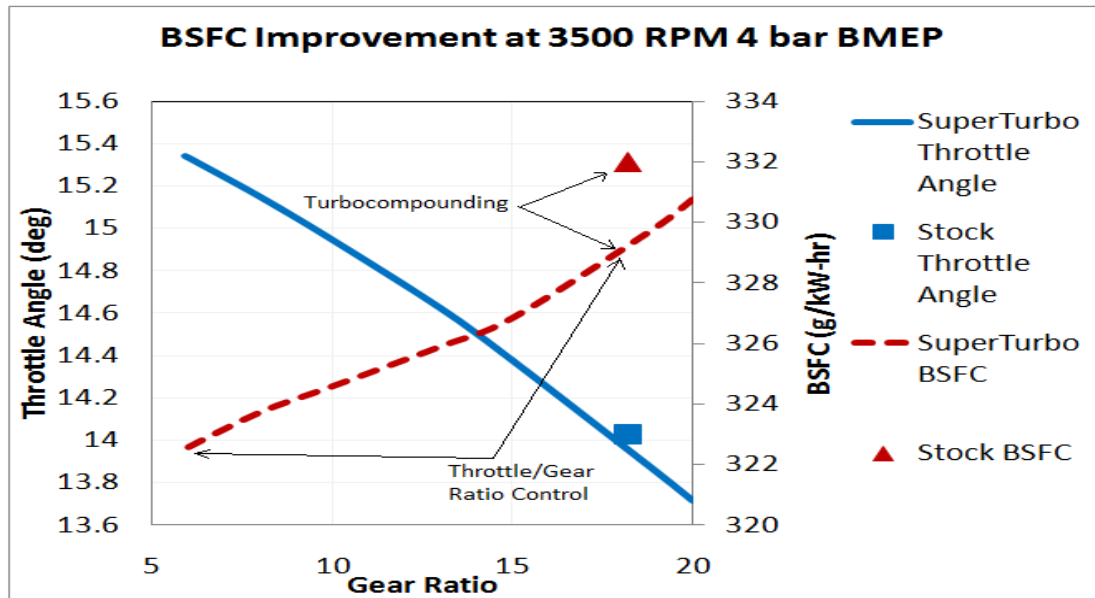


Figure 43: Brake specific fuel consumption improvement over stock engine as low load case where BMEP contribution from the turbocompounding as shown in Figure 3.8. This contribution means the in-cylinder power requirement is reduced resulting in overall fuel savings. BMEP contribution is obtained by calculating in-cylinder BMEP values as listed below and subtracting that number from the overall BMEP for the entire engine system reported by GT-Power.

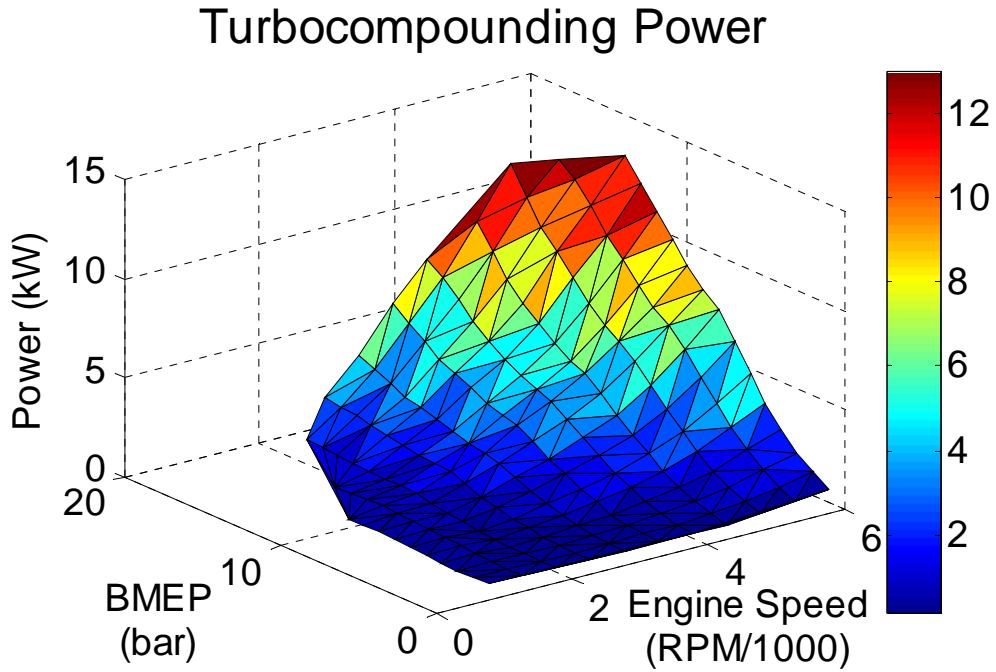


Figure 46: Power transferred from SuperTurbo to Crankshaft

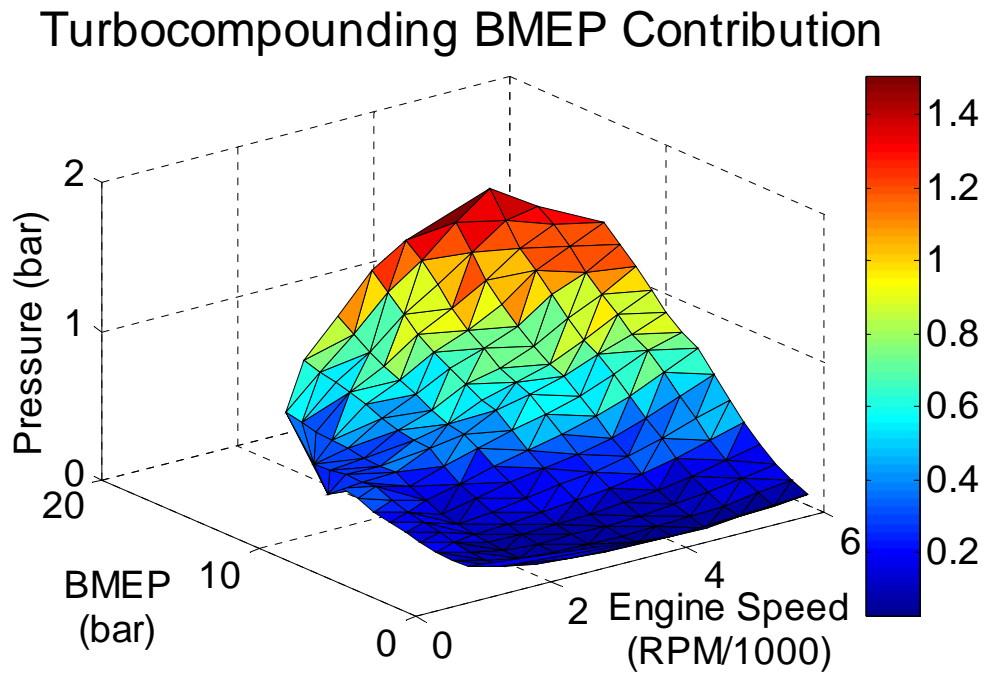


Figure 45: BMEP contribution from Turbocompounding

$$\text{BMEP} = \text{IMEP}(360) + \text{PMEP} - \text{FMEP}$$

where,

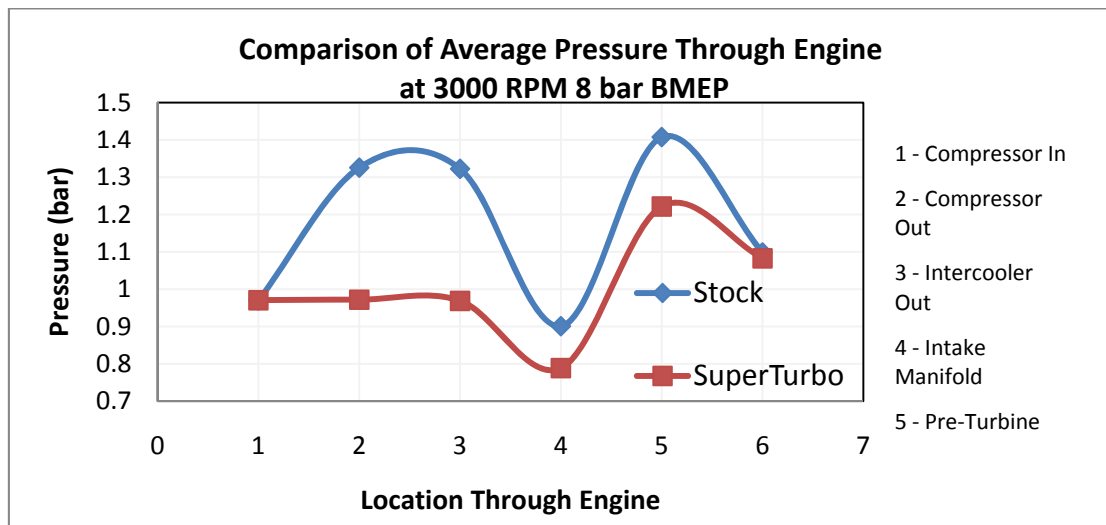
BMEP = Brake Mean Effective Pressure

IMEP(360) = Indicated Mean Effective Pressure (during compression/combustion stroke)

PMEP = Pumping Mean Effective Pressure

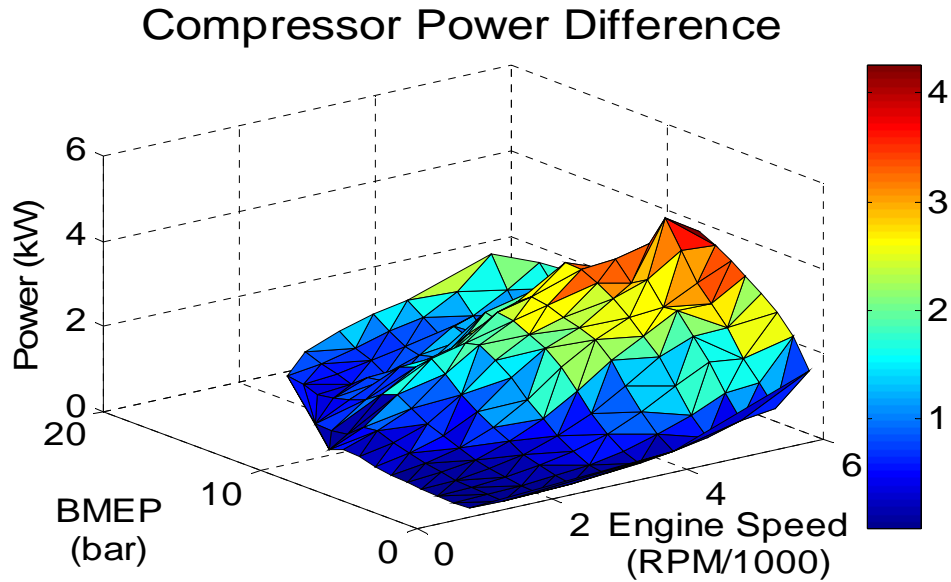
FMEP = Friction Mean Effective Pressure

The mid to lower load region of the map is improved by the reduced use of the compressor when it is not needed as illustrated in Figure 3.9. The mid region of the map saves



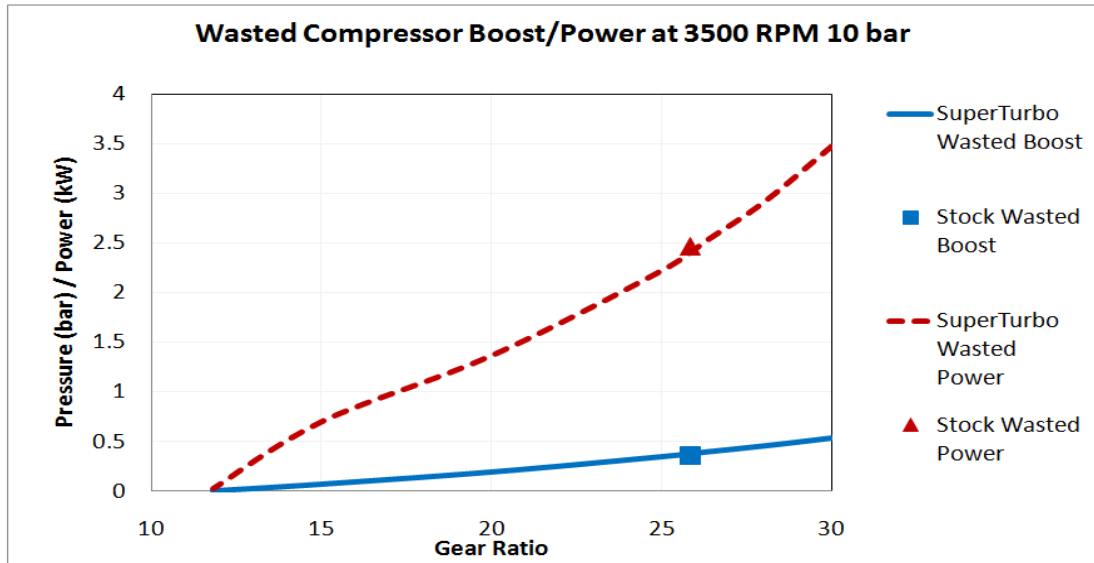
**Figure 47: Pressure at different locations through the engine for 3000 RPM 8 bar BMEP**

up to three or four kilowatts that are typically wasted by the compressor due to the SuperTurbo control of the gear ratio as illustrated in Figure 3.10. The difference is due to the ability of the superturbo to control the pressure ratio of the compressor. At low loads where the intake manifold pressure is below atmospheric the pressure ratio is controlled to be just above one where at higher loads the throttle is wide open and the gear ratio is controlled to give the required manifold pressure. This power difference correlates well with Figure 3.2 above showing areas where boost pressure is not needed. This reduction in power also corresponds to a reduction in in-cylinder power requirement for the same brake power.



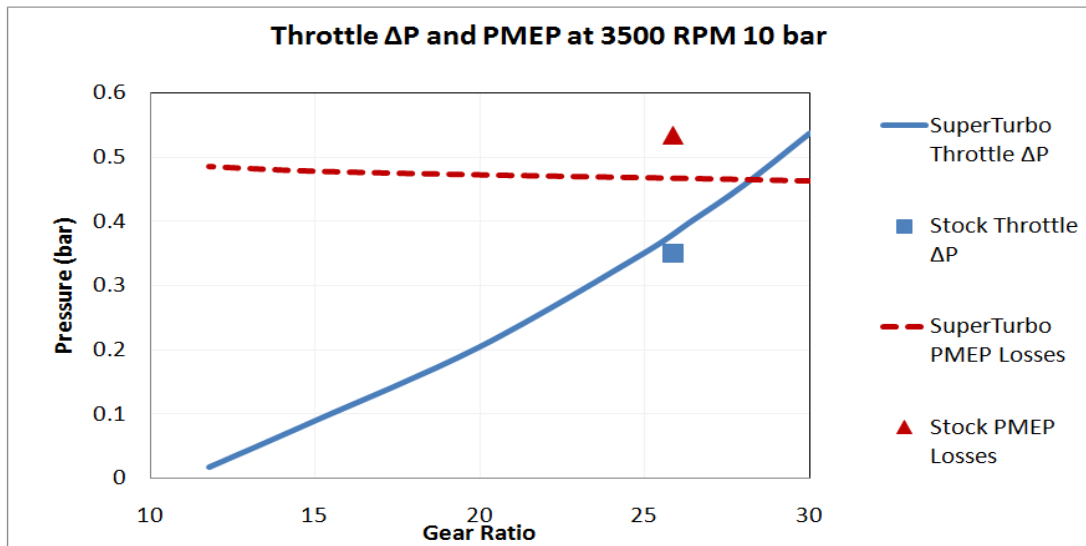
**Figure 48: Power transferred from SuperTurbo to Crankshaft**

While the wasted compressor boost pressure and power goes down with lower gear ratio as shown in Figure 3.11, the PMEP rises slightly with decreasing gear ratio even though pressure drop across the throttle is decreasing as shown in Figure 3.12. From the stock configuration to the superturbo configuration there is a reduction in pumping losses, but as the gear ratio is reduced the pumping losses gradually creep higher. The reduction in pumping losses at this location comes from the reduced backpressure of the improved turbine design as illustrated in Figure 3.13. With the stock gear ratio imposed there are additional pumping losses on the intake side due to the slightly lower manifold pressure. These losses are made up for by the higher reduction in pumping losses on the exhaust side. As the gear ratio is reduced the exhaust pressure drops, but not as much as the intake pressure drops which causes a very slight increase in pumping losses. These pumping losses are insignificant compared to the reduced compressor power losses shown above leading to overall BSFC improvement.

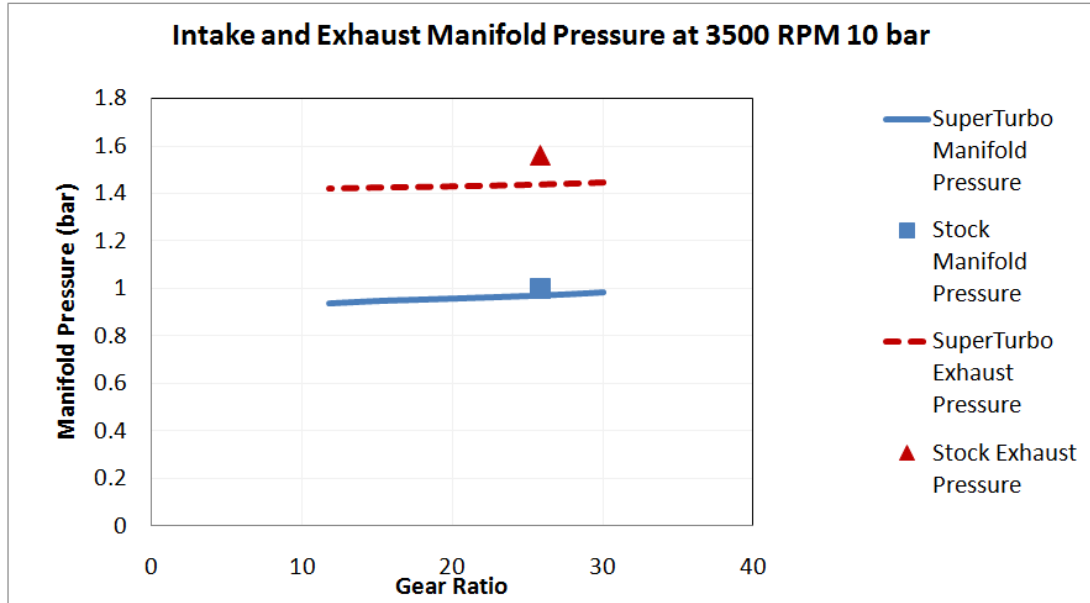


**Figure 50: Wasted boost pressure and compressor power as a function of SuperTurbo gear ratio**

A comparison of the log pressure vs. volume (P-V) plots shows the difference in intake and exhaust pressures in Figure 3.14. For the 3500 RPM 4 bar BMEP case the savings from lower exhaust back pressure clearly makes up for the lower intake manifold pressure. For the higher load case of 15 bar BMEP the pumping losses are higher than stock as illustrated in Figure 3.15. For this case the backpressure is close to stock, but the intake pressure is reduced enough to



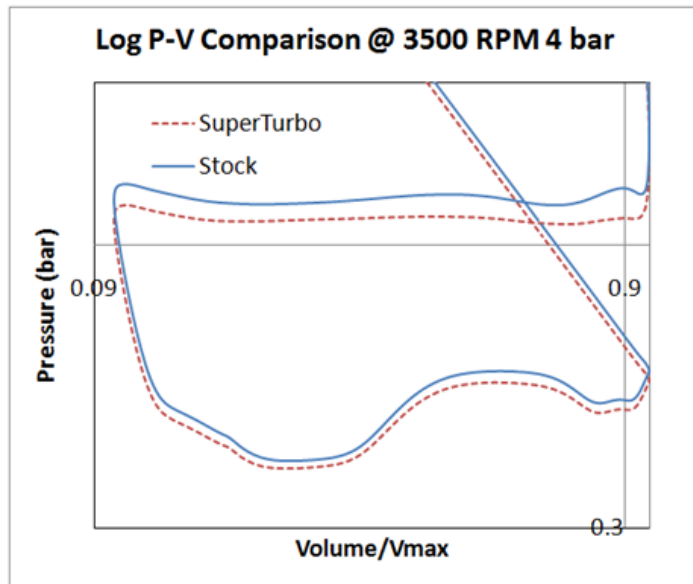
**Figure 49: While the pressure drop across the throttle reduces with gear ratio, the pumping losses increase slightly**



**Figure 51: The reduced backpressure more than compensates for the increased pumping losses from the reduced intake manifold pressure**

cause an overall increase in pumping losses. Although the pumping losses are higher compared to stock, they are insignificant compared to the turbocompounding benefit.

The intake manifold pressure is lower for the SuperTurbo configuration over the entire operating range as shown in Figure 3.16. The entire area has a reduction in pressure of about 2-



7%. This is to be expected since the cylinder does not need as much air to create the same overall power in the system. While the new turbine design allows reduced backpressure for much of the operating range as shown in Figure 3.17 the higher speed

**Figure 52: Log P-V plot comparing pumping losses at 3500 RPM 4**



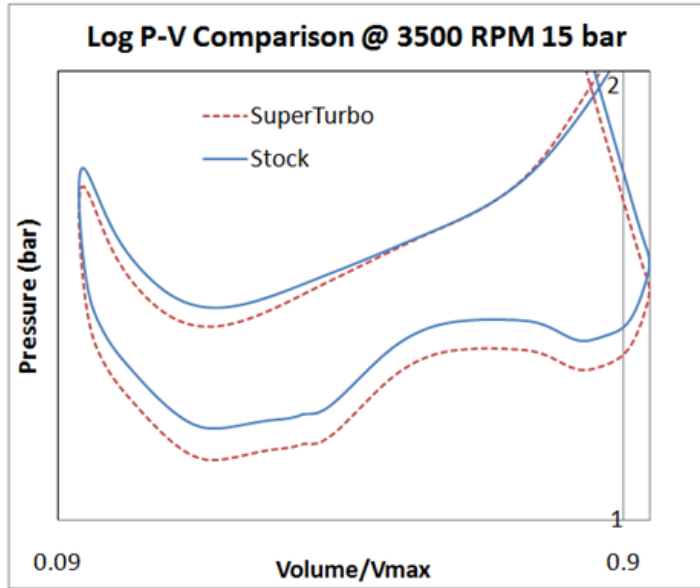


Figure 53: Log P-V plot comparing pumping losses at 3500 RPM 15

and load area has higher backpressure due to the wastegate being closed for turbocompounding. These pressure differences correlate to lower pumping losses in the mid range of the engine while the high load and high engine speed area of the engine

experiences higher pumping losses compared to stock as illustrated in Figure 3.18. The low load and speed area shows almost no difference. The overall gear ratio used stays under 30 for most of the map except for the low speed high load region as Figure 3.19 shows.

Another source of improvement not realized with the modeling relates to the intake air

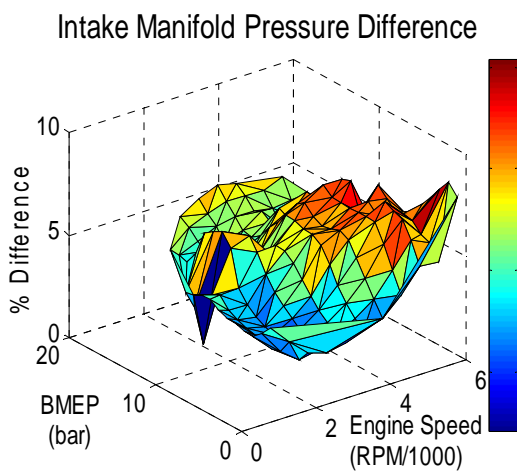


Figure 54: Difference in intake manifold pressure calculated as stock pressure minus SuperTurbo pressure so (+) indicates stock pressure is greater

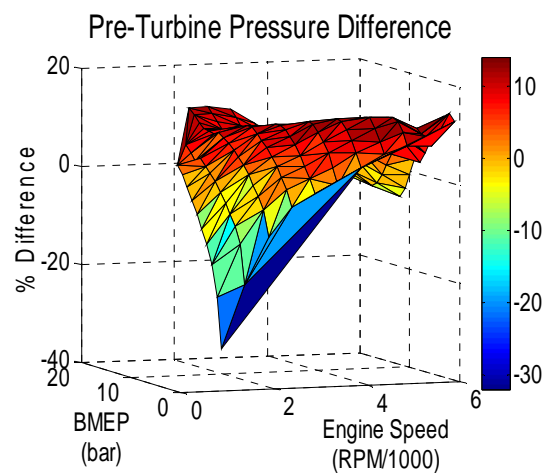
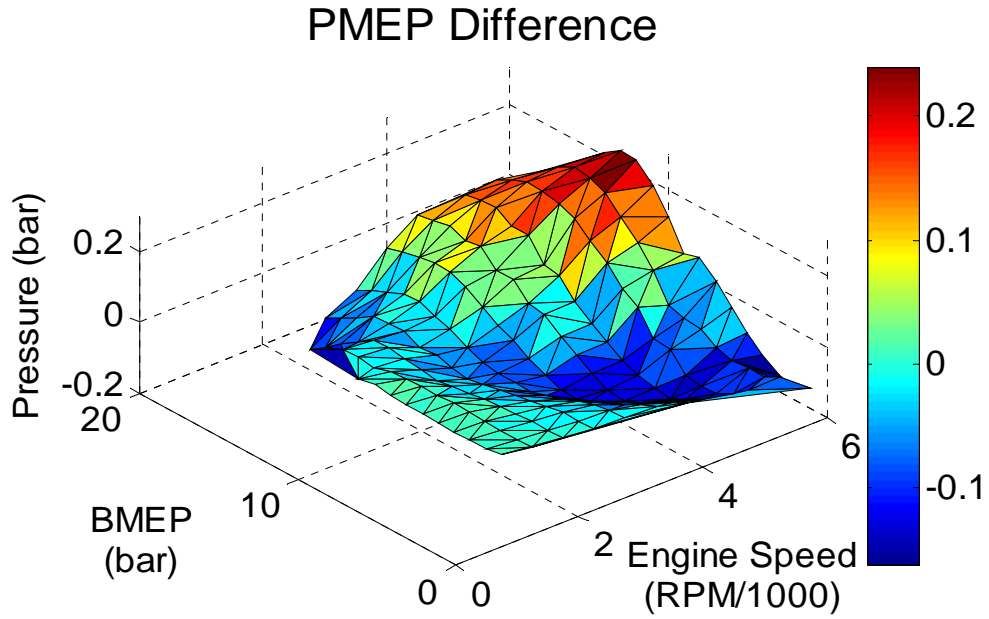
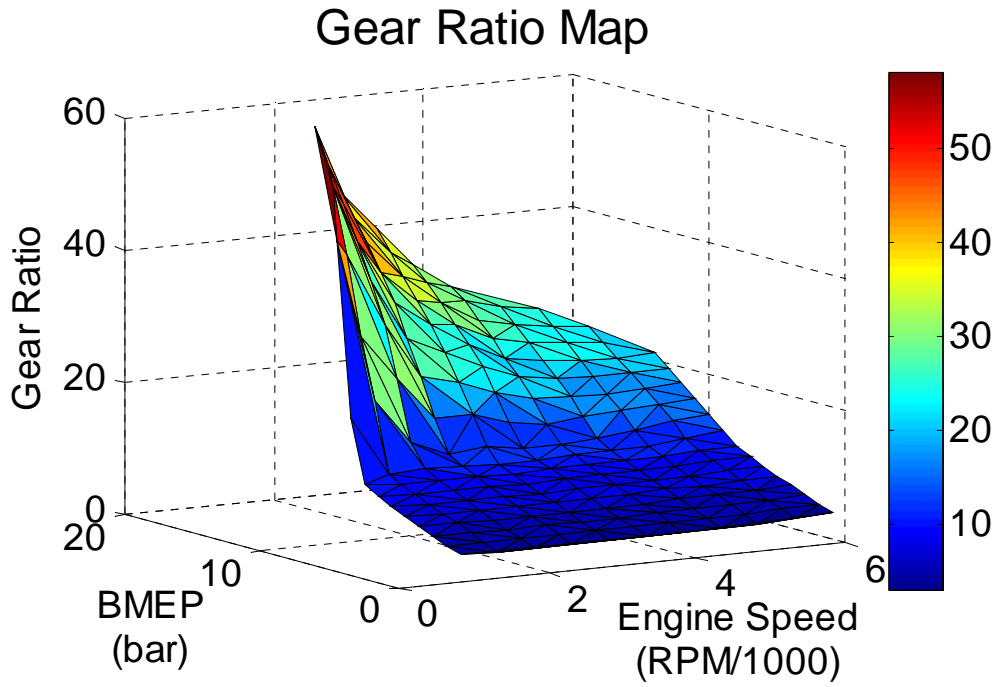


Figure 55: Difference in pre-turbine pressure calculated as stock pressure minus SuperTurbo pressure so (+) indicates stock pressure is greater



**Figure 57: Stock PMEP minus SuperTurbo PMEP**

temperature. The model does a poor job of estimating the temperature rise across the compressor, but the differences can be estimated with the use of test data. The lower load region of the map where the compressor is controlled to produce no boost should have almost



**Figure 56: Overall gear ratio from turbine shaft to crankshaft**

no temperature rise compared to the stock configuration. This results in lower intake manifold temperatures which provide denser in-cylinder air meaning less pressure is required.

The model results help solidify the understanding of how the SuperTurbo is able to benefit a reciprocating engine over the entire steady state operating range. The turbocompounding advantage goes up with higher load and engine speed since that is the area where most of the additional exhaust energy is passed through the wastegate. The mid to lower load range of the engine is able to realize more benefit due to the control of the compressor speed. Instead of creating boost that is not needed to prevent turbo lag boost is able to be created on demand at any given operating point with the use of the SuperTurbo. The mid range also benefits from reduced pumping losses. Even though the higher load and speed range has higher pumping losses compared to stock, the benefits from turbocompounding more than make up for the pumping losses and is able to supplement some of the in-cylinder BMEP requirement.

## **CHAPTER 4: SUMMARY AND CONCLUSION**

### **4.1 Summary**

A modeling project was carried out on a Volkswagen engine model using GT-Power. The addition of a SuperTurbo to the model was used to predict potential BSFC improvements. The SuperTurbo acts as a supercharger for the engine giving the intake manifold boost when it is needed and provides turbocompounding when excess boost is available. The stock engine model was used that Volkswagen had already correlated with test data for a wide open throttle configuration. Additional test data was used to correlate the data over the entire range of operating points. The SuperTurbo model developed by SwRI was added to the model and the original turbine model was replaced by a simplified version of a turbine model to represent characteristics of a new turbine design for use in the superturbo configuration.

It was predicted that the improvements would come from two sources. One source of improvement is where the turbine wastegate is open. This wasted energy was available to offset in-cylinder power with the use of turbocompounding. The second source of improvement comes from the reduced use of the compressor at low loads where boost is not needed. The modeling results confirmed the areas of improvement. The actual magnitude of improvements will depend on transmission efficiencies yet to be determined.

## 4.2 Conclusion and Future Work

- The original model required modification for accurate part load simulations.
- The improved model is able to match stock data with acceptable accuracy due to imposing engine timing parameters from the test data as well as modifying the FMEP model and throttle model.
- The SuperTurbo simulations resulted in significant BSFC benefits.
- The area of operation where the stock engine wastegate is normally open shows improvement due to utilization of excess exhaust energy through turbocompounding with a maximum of 6.7% improvement at 5000 RPM; 17 bar BMEP although the maximum amount of turbocompounding is 12.95 kW at 6000 RPM; 14 bar BMEP.
- The area of operation where boost pressure is limited when it is not needed also shows improvement due to the minimization of wasted compressor power with the largest power savings being 4.3 kW at 5500 RPM; 8 bar BMEP which coincides with the highest point for wasted boost pressure.
- The simulations showed approximate magnitudes of possible improvements with an assumed transmission mechanical efficiency of 80%.
- The actual magnitudes of improvements will depend heavily on the final efficiency of the turbocompounding action of the transmission between the turbine shaft and the crankshaft.

Possibilities of future work consist of further improvements to the current configuration by optimizing the timing parameters. This will require knowledge about the limitations affecting the current timing parameters. Once more information is known about the parasitic losses and

mechanical efficiency of the final transmission it can be included into the model to show more realistic improvements.

Now that steady state results have been modeled these can be incorporated into transient models that are able to benefit more from the supercharging function of the SuperTurbo. The combination of transient and steady state modeling can then be used to determine fuel saving benefits over prescribed drive cycles. With a working model capable of both transient and steady state operation, control strategies can be developed and tested before being implemented on hardware.

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## Appendix A: Volkswagen Test Data

	DREH			PME	ED	EE	EB	EK	NW_E	VPBS	BE98	ZZP
	engine speed	Torque	Power	bmeP	injection time	injection end	injection start	injection duration	intake cam shaft position	fuel consumption	spec. fuel consumption	ignition timing
Index	[1/min]	[Nm]	[kW]	[bar]	[ms]	[%kWOT]	[%kWOT]	[%kW]	[%kW]	[l/h]	[g/kWh]	[%kWOT]
1	1010	16.2	1.72	1.02	0.95	304	310	6	-1	1.25	552.8	20
2	1009	31.2	3.30	1.96	1.31	304	312	8	-4	1.71	394.1	20
3	1006	47.4	5.00	2.98	1.64	307	317	10	-9	2.22	337.3	19
4	1006	64.9	6.84	4.08	1.88	305	316	11	-10	2.71	301.2	17
5	1009	80.2	8.48	5.04	2.11	299	312	13	-10	3.18	285	15
6	1006	95.0	10.01	5.97	2.28	295	309	14	-10	3.59	272.6	14
7	1004	111.4	11.71	7	2.43	294	309	15	-9	4.11	266.7	11
8	1018	128.3	13.68	8.06	2.69	292	308	16	-10	4.69	260.6	8
9	1011	141.6	15.00	8.9	2.93	286	306	18	-7	5.14	260.5	5
10	1010	158.8	16.80	9.98	3.17	281	300	19	-7	5.63	263.5	3
11	1011	162.3	17.19	10.2	3.29	278	296	20	-6	6.12	270.5	2
12	1500	15.4	2.43	0.97	0.94	309	317	8	0	1.53	601.9	23
13	1500	31.7	4.98	1.99	1.29	307	319	12	-16	2.67	404.4	22
14	1500	47.3	7.43	2.97	1.52	304	318	14	-28	3.26	331.1	22
15	1499	62.5	9.82	3.93	1.77	302	318	16	-35	3.91	300	21
16	1501	78.5	12.33	4.93	1.97	300	318	18	-35	4.55	278.4	20
17	1501	95.8	15.06	6.02	2.16	298	317	19	-36	5.3	265.4	19
18	1499	112.0	17.59	7.04	2.43	297	319	22	-31	6.05	259.8	15
19	1500	125.5	19.71	7.89							258.2	
20	1500	128.4	20.18	8.07	2.67	297	321	24	-26	6.89	257.9	16
21	1500	143.9	22.60	9.04	2.76	291	316	25	-15	7.78	259.7	11
22	1499	176.0	27.63	11.06	3.26	277	306	29	-9	9.36	255.8	7
23	1498	213.3	33.46	13.4	4.09	269	306	37	-5	11.95	269.5	0
24	2000	17.0	3.67	1.07	0.86	309	319	10	-34	2.71	579.8	17
25	2000	32.0	6.70	2.01	1.28	304	319	15	-36	3.53	400.1	29
26	2000	47.0	9.83	2.95	1.23	304	319	15	-38	4.32	333.2	28
27	2000	63.5	13.30	3.99	1.46	301	319	18	-39	5.3	302.9	26
28	2000	80.4	16.83	5.05	1.66	299	319	20	-38	6.25	282.2	24
29	2000	95.8	20.07	6.02	1.86	296	318	22	-38	7.13	269.8	23
30	2000	111.6	23.37	7.01	2.18	290	316	26	-37	8.05	261.8	21
31	2000	127.5	26.70	8.01	2.3	284	312	28	-34	9.04	257.3	20
32	2000	145.3	30.43	9.13	2.53	278	308	30	-30	10.2	254.5	18
33	2000	161.7	33.87	10.16	2.83	274	308	34	-23	11.24	252.1	14
34	2000	177.1	37.10	11.13	3.03	272	308	36	-18	12.18	249.5	12
35	2000	190.7	39.93	11.98	3.2	274	312	38	-14	12.98	246.8	9
36	2000	207.2	43.40	13.02	3.48	274	316	42	-10	14.11	246.9	9
37	2000	243.5	51.00	15.3	4.03	273	321	48	-9	17.09	254.5	4
38	2000	254.8	53.37	16.01	4.48	261	315	54	-6	18.09	257.3	2
39	2000	280.1	58.67	17.6	5.31	243	307	64	-3	21.28	275.6	-3
40	2500	16.7	4.38	1.05	0.82	311	323	12	-35	3.52	613.6	31
41	2500	32.6	8.54	2.05	1.03	308	323	15	-36	4.61	408.8	30
42	2500	47.9	12.54	3.01	1.24	305	324	19	-38	5.67	343.4	29
43	2500	63.0	16.71	4.01	1.47	300	322	22	-39	6.0	309.2	27
44	2500	80.9	21.17	5.08	1.77	293	320	27	-39	8.01	287.3	25
45	2500	96.6	25.29	6.07	1.89	290	318	28	-38	9.14	274.3	23
46	2500	111.4	29.17	7	2.11	284	316	32	-37	10.17	265	22
47	2500	129.1	33.79	8.11	2.38	277	313	36	-34	11.44	257.3	19
48	2500	143.6	37.58	9.02	2.63	273	312	39	-31	12.58	254.2	16
49	2500	160.0	41.88	10.05	2.77	271	313	42	-23	13.96	253.2	13
50	2500	173.8	45.50	10.92	2.96	268	312	44	-18	14.91	248.9	12
51	2500	189.6	49.63	11.91	3.18	267	315	48	-15	16.04	245.3	13
52	2500	206.7	54.13	12.99	3.41	266	317	51	-10	17.44	244.6	10
53	2500	224.9	58.88	14.13	3.73	264	320	56	-9	18.95	244.4	8
54	2500	239.5	62.71	15.05	3.99	262	322	60	-9	20.45	247.8	5
55	2500	271.4	71.04	17.05	4.81	240	312	72	-1	24.31	259.9	2
56	3002	16.6	5.20	1.04	0.86	312	327	15	-35	4.44	646.4	32
57	3000	35.0	11.00	2.2	1.07	309	328	19	-36	6.23	429.3	31
58	3002	47.6	14.96	2.99	1.31	304	326	24	-38	7.03	357.2	29
59	3002	63.7	20.01	4	1.47	299	325	26	-39	8.36	317	28
60	3002	81.5	25.62	5.12	1.78	288	320	32	-38	9.85	291.9	25
61	3002	97.1	30.52	6.1	1.94	284	319	35	-38	11.23	279.4	23
62	3002	111.1	34.92	6.98	2.13	279	317	38	-37	12.48	271.4	22
63	3002	127.3	40.03	8	2.31	277	319	42	-34	13.88	263.5	20
64	3002	144.8	45.53	9.1	2.63	272	319	47	-29	15.43	257.4	20
65	3002	159.8	50.23	10.04	2.73	268	317	49	-24	16.77	253.4	19
66	3000	179.5	56.40	11.28	3	263	317	54	-17	18.38	247.5	17
67	3000	193.9	60.90	12.18	3.24	259	317	58	-5	19.75	246.4	15
68	3000	209.3	65.75	13.15	3.52	255	318	63	-4	21.2	244.8	12
69	3000	222.5	69.90	13.98	3.82	249	318	69	-4	22.7	246.6	11
70	3000	239.5	75.25	15.05	4.17	243	318	75	-4	24.86	250.8	9
71	3000	253.9	79.75	15.95	4.43	238	318	80	-3	27.12	258.3	7
72	3000	269.3	84.60	16.92	4.96	233	318	89	-3	29.87	269.2	5
73	3500	15.4	5.66	0.97	0.9	318	337	19	-6	5.24	702.6	30
74	3500	32.0	11.79	2.01	1.02	315	336	21	-6	6.91	446.6	28
75	3500	47.7	17.50	3	1.29	310	337	27	-6	8.44	366.7	28
76	3500	63.7	23.33	4	1.49	302	333	31	-6	10.1	328.8	26
77	3500	81.3	29.81	5.11	1.79	291	329	38	-5	11.9	303	25
78	3500	96.9	35.53	6.09	1.95	285	326	41	-5	13.6	291	23
79	3500	111.7	40.95	7.02	2.18	276	322	46	-5	15.02	278.6	22
80	3500	127.3	46.67	8	2.44	267	318	51	-5	16.58	270	20
81	3500	143.1	52.44	8.99	2.57	264	318	54	-5	18.11	262.3	20
82	3500	161.4	59.15	10.14	2.81	260	319	59	-5	19.99	256.6	19
83	3500	175.2	64.23	11.01	3.03	255	319	64	-4	21.49	254.1	17
84	3500	192.7	70.64	12.11	3.29	249	318	69	-4	23.34	251.1	15
85	3500	209.3	76.71	13.15	3.57	244	319	75	-4	25.39	251.4	14
86	3500	221.5	81.20	13.92	3.78	239	318	79	-4	26.98	252.4	13
87	3500	241.6	88.55	15.18	4.2	230	318	88	-1	29.85	266	10
88	3500	256.4	93.98	16.11	4.71	219	318	99	-1	32.76	264.9	9
89	3499	276.1	101.18	17.35	5.06	212	318	106	-1	36.83	276.4	6

	DREH			PME	ED	EE	EB	EK	NW_E	VPBS	BE98	ZZP
	engine speed	Torque	Power	brnep	injection time	injection end	injection start	injection duration	intake cam shaft position	fuel consumption	spec. fuel consumption	ignition timing
Index	[1/min]	[Nm]	[kW]	[bar]	[ms]	[°KWvOT]	[°KWvOT]	[°KW]	[°KW]	[l/h]	[g/kWh]	[°KWvOT]
90	4000	16.9	7.07	1.06	0.89	317	338	21	-5	6.51	697.5	30
91	4000	31.8	13.33	2	1.09	312	338	26	-5	8.25	469.7	29
92	4000	48.7	20.40	3.06	1.34	306	338	32	-5	10.32	384.6	28
93	4000	64.3	26.93	4.04	1.55	301	338	37	-4	12.08	341.1	26
94	4000	80.2	33.60	5.04	1.73	297	339	42	-5	13.93	314.8	25
95	4000	94.4	39.53	5.93	1.93	290	336	46	-5	15.44	296.6	23
96	4000	111.2	46.60	6.99	2.15	286	338	52	-5	17.54	285.7	23
97	4000	126.4	52.93	7.94	2.4	281	339	58	-4	19.17	274.9	20
98	4000	143.1	59.93	8.99	2.63	271	334	63	-4	21.05	266.9	20
99	4000	161.1	67.47	10.12	2.94	257	328	71	-4	23.14	260.5	19
100	4000	174.1	72.93	10.94	3.11	250	325	75	-4	24.91	259.4	18
101	4000	193.4	81.00	12.15	3.4	238	320	82	-4	27.55	258.3	16
102	4000	210.7	88.27	13.24	3.61	232	319	87	-4	29.99	258.1	15
103	4000	221.5	92.80	13.92	3.87	225	318	93	-4	32.01	262	13
104	4000	241.9	101.33	15.2	4.54	209	318	109	-4	35.71	267.7	11
105	4000	253.9	106.33	15.95	4.81	202	317	115	-3	39.12	279.4	8
106	4000	272.0	113.93	17.09	5.67	181	317	136	-1	45.04	300.1	7
107	4000	284.7	119.27	17.89	6.02	179	323	144	0	48.86	311.2	6
108	4500	15.8	7.43	0.99	0.91	313	338	25	-5	7.61	775.5	28
109	4500	32.6	15.38	2.05	1.19	306	338	32	-5	9.76	481.6	26
110	4500	48.5	22.88	3.05	1.35	301	337	36	-5	11.94	396	25
111	4500	64.6	30.45	4.06	1.56	296	338	42	-5	14.05	350.1	24
112	4500	79.6	37.50	5	1.73	291	338	47	-5	16.02	324.4	24
113	4500	98.0	46.20	6.16	2.1	282	339	57	-4	18.4	302.7	22
114	4500	112.4	52.95	7.06	2.25	277	338	61	-5	20.27	290.7	21
115	4500	128.0	60.30	8.04	2.5	269	336	67	-5	22.57	284.2	20
116	4500	139.9	65.93	8.79	2.67	262	334	72	-5	24.26	279.4	19
117	4500	162.3	76.50	10.2	3.02	245	327	82	-5	27.44	272.3	19
118	4500	177.5	83.63	11.15	3.31	238	327	89	-4	29.66	269.3	18
119	4500	191.9	90.45	12.06	3.43	234	327	93	-5	31.98	268.5	17
120	4500	209.4	98.70	13.16	3.82	223	326	103	-4	35.25	271.2	14
121	4500	223.9	105.53	14.07	4.19	208	321	113	-3	38.8	279.3	12
122	4500	237.9	112.13	14.95	4.59	194	318	124	-3	42	284.5	10
123	4500	253.9	119.63	15.95	5.13	195	333	138	-1	46.61	296	7
124	4501	270.6	127.53	17	5.92	191	351	160	0	55.21	328.9	4
125	5000	16.2	8.50	1.02	0.96	309	338	29	-5	9.09	815.8	27
126	5000	30.1	15.75	1.89	1.19	302	338	36	-5	11.13	535.9	26
127	5000	50.1	26.25	3.15	1.49	296	341	45	-5	14.09	408	24
128	5000	66.4	34.75	4.17	1.7	287	330	51	-5	16.5	360.5	24
129	5000	79.6	41.67	5	1.85	282	338	56	-4	18.31	333.7	23
130	5000	94.4	49.42	5.93	2.05	276	338	62	-5	20.72	318.6	22
131	5000	114.3	59.83	7.18	2.42	266	339	73	-4	24.21	307.4	20
132	5000	128.8	67.42	8.09	2.68	258	338	80	-5	27.07	305.1	21
133	5000	145.0	75.92	9.11	2.93	250	338	88	-4	29.88	298.8	21
134	5000	160.4	84.00	10.08	3.22	243	340	97	-4	32.41	293.1	19
135	5000	177.8	93.08	11.17	3.44	242	345	103	-4	35.46	289.4	19
136	5000	195.8	102.50	12.3	3.97	238	357	119	-4	38.72	287.1	16
137	5000	208.8	109.33	13.12	4.11	231	354	123	-4	41.79	290.3	14
138	5000	221.9	116.17	13.94	4.4	224	356	132	-4	45.48	297.3	12
139	5000	237.1	124.17	14.9	4.83	212	357	145	-3	49.33	301.8	11
140	5000	255.8	133.92	16.07	5.41	195	357	162	-1	55.98	317.4	9
141	5000	265.2	138.83	16.66	6.14	173	357	184	-1	61.06	334	6
142	5500	16.1	9.26	1.01	0.99	313	346	33	-4	10.48	861	28
143	5500	34.9	20.08	2.19	1.23	305	346	41	-5	13.52	511	25
144	5500	50.0	28.78	3.14	1.47	299	348	49	-4	15.98	421.4	24
145	5500	61.1	35.20	3.84	1.67	296	351	55	-4	17.83	384.3	23
146	5500	78.0	44.92	4.9	1.99	292	358	66	-4	20.87	352.8	22
147	5500	95.8	55.18	6.02	2.19	283	355	72	-4	24.33	334.7	22
148	5500	114.1	65.73	7.17	2.52	272	355	83	-3	28.01	323.8	21
149	5500	129.2	74.43	8.12	2.76	267	358	91	-3	30.95	315.7	21
150	5500	143.9	82.87	9.04	3.07	261	362	101	-3	34.1	312.5	20
151	5500	161.1	92.77	10.12	3.33	256	366	110	-3	37.73	309	19
152	5500	176.8	101.84	11.11	3.66	245	366	121	-3	41.15	306.7	17
153	5500	190.7	109.82	11.98	3.91	237	366	129	-3	44.78	309.8	16
154	5500	207.4	119.44	13.03	4.25	226	366	140	-3	49.42	314.3	15
155	5500	217.4	125.22	13.66	4.54	216	366	150	-3	53.34	323.5	13
156	5500	243.3	140.16	15.29	5.54	180	363	183	-1	66.26	359	10
157	6000	15.6	9.80	0.98	1.11	308	348	40	-1	12.61	974.2	28
158	6001	32.8	20.60	2.06	1.34	299	347	48	0	15.85	585.3	25
159	6000	49.3	31.00	3.1	1.58	298	355	57	0	19.26	472.1	23
160	6000	63.2	39.70	3.97	1.83	297	363	66	0	21.98	420.6	23
161	6000	80.7	50.70	5.07	2.15	291	368	77	-1	25.46	381.3	22
162	6000	95.7	60.10	6.01	2.39	287	373	86	-1	28.56	360.6	21
163	6000	113.3	71.20	7.12	2.7	277	374	97	-2	32.29	344.6	21
164	6000	127.8	80.30	8.03	2.95	269	375	106	-2	35.96	339.9	20
165	6000	142.8	89.70	8.97	3.26	259	376	117	-2	39.6	335.2	19
166	6000	158.8	99.80	9.98	3.53	250	377	127	-2	43.73	332.9	19
167	6000	180.0	113.10	11.31	4.07	231	378	147	-2	49.33	331.2	16
168	6000	188.9	118.70	11.87	4.25	223	376	153	-2	52.23	334.2	15
169	6000	206.4	129.70	12.97	4.93	199	376	177	-2	59.8	350	13
170	6001	224.4	141.02	14.1	5.56	175	375	200	-1	70.94	382.1	12

Index	TLVV	TLNV	TLNK	TSAU	TAK1	TAK2	TAK3	TAK4	TAVT	TANT	TAVK	TANK
	before compressor T [°C]	after compressor T [°C]	after intercooler T [°C]	manifold temperature [°C]	exhaust manifold T1 [°C]	exhaust manifold T2 [°C]	exhaust manifold T3 [°C]	exhaust manifold T4 [°C]	before turbine T [°C]	after turbine T [°C]	before catalyst T [°C]	after catalyst T [°C]
1	22	30	10	26	410	404	382	391	378	287	290	320
2	21	28	17	25	449	436	423	424	383	308	308	359
3	22	30	17	26	459	458	453	448	412	334	336	383
4	22	30	21	26	457	469	466	458	446	374	378	410
5	22	31	23	27	460	479	477	460	474	397	400	439
6	23	32	24	28	462	490	489	465	497	419	421	468
7	23	34	26	30	469	506	499	476	525	450	450	498
8	24	36	26	30	480	521	526	492	556	480	480	527
9	24	38	26	30	491	534	540	500	578	501	503	541
10	24	40	26	28	500	547	552	514	601	521	523	563
11	25	41	28	29	501	549	552	511	602	522	522	533
12	22	29	16	25	505	480	474	490	458	367	366	398
13	23	30	18	25	516	508	518	502	477	393	393	423
14	24	31	18	26	507	516	525	502	504	431	433	437
15	24	33	16	27	504	518	518	494	541	461	461	469
16	24	35	21	27	515	532	528	508	567	489	488	502
17	24	37	22	28	521	544	538	517	587	505	506	529
18	25	41	24	28	538	569	563	543	619	535	535	560
19												
20	25	45	28	30	550	595	588	561	645	561	562	601
21	25	48	27	30	579	613	621	590	680	590	592	631
22	25	52	30	31	581	643	651	600	726	651	651	683
23	27	72	33	32	616	692	695	642	790	655	656	667
24	21	30	8	26	546	530	519	525	481	387	387	431
25	22	31	14	25	586	578	565	568	528	460	475	494
26	23	33	17	26	588	588	571	573	572	504	504	533
27	24	37	21	27	592	595	580	577	607	526	526	564
28	23	39	23	27	596	606	590	585	634	548	549	590
29	23	42	25	28	603	619	604	598	655	570	572	610
30	24	47	25	29	612	631	624	611	677	588	586	638
31	24	51	26	29	622	641	644	628	701	609	609	654
32	25	56	27	29	634	652	665	642	727	639	639	682
33	24	59	29	30	641	666	679	647	746	659	660	700
34	25	62	30	30	642	672	685	650	758	671	672	716
35	25	62	30	31	649	685	696	657	778	691	689	732
36	25	64	30	31	654	698	713	666	799	710	715	749
37	25	82	32	32	690	753	760	705	862	751	748	763
38	25	88	33	32	695	758	761	713	866	749	750	754
39	25	109	33	33	719	783	779	737	891	752	752	743
40	22	32	8	24	602	588	580	582	544	452	449	510
41	22	33	16	25	635	639	626	618	599	530	531	562
42	22	37	19	26	633	643	635	620	634	562	564	592
43	23	42	19	26	642	649	648	623	665	582	583	623
44	24	47	22	27	648	658	656	628	687	599	600	651
45	24	52	26	27	656	673	669	642	711	620	620	674
46	24	55	26	28	658	681	676	651	724	636	638	689
47	25	60	26	29	670	694	689	667	746	658	659	708
48	25	63	28	30	679	705	700	678	763	674	676	724
49	25	64	30	31	696	721	721	694	791	704	702	752
50	25	64	30	31	696	727	723	695	797	712	715	763
51	25	64	32	33	702	738	734	707	816	729	732	779
52	26	65	32	33	715	749	747	717	838	748	751	796
53	26	70	34	34	732	769	765	730	864	767	769	810
54	26	74	34	34	737	784	779	741	880	778	783	807
55	25	96	36	36	757	808	796	763	904	783	788	786
56	24	36	12	25	687	683	679	665	650	557	557	587
57	24	39	13	24	690	695	684	675	667	586	582	618
58	24	43	20	27	692	699	689	682	693	614	614	651
59	25	49	23	28	699	706	697	687	719	631	630	672
60	25	55	25	29	706	715	704	692	742	656	657	703
61	26	60	26	30	716	728	714	700	763	679	678	723
62	26	64	26	29	721	736	724	708	777	690	691	741
63	27	67	28	31	730	749	740	718	797	708	708	759
64	27	68	29	32	738	756	747	728	813	726	725	778
65	26	63	33	33	738	761	755	734	827	742	741	796
66	26	63	33	34	742	768	761	742	842	761	760	809
67	26	63	34	34	739	767	765	744	849	774	774	829
68	26	63	35	35	745	776	769	750	860	787	789	828
69	26	69	35	36	751	787	779	761	873	791	793	822
70	26	75	37	37	759	795	791	768	886	795	795	815
71	26	84	39	38	764	804	796	773	893	790	791	804
72	26	94	40	40	774	814	805	782	905	791	790	797
73	24	36	18	25	714	701	698	682	691	610	616	644
74	24	39	22	26	722	733	725	702	709	643	646	675
75	25	45	24	27	734	742	736	723	738	666	666	710
76	25	51	25	28	739	755	746	734	766	686	688	733
77	25	58	26	29	742	761	753	738	787	705	707	761
78	26	63	28	30	745	765	758	745	803	719	720	780
79	26	67	29	31	751	769	764	749	814	728	731	787
80	26	66	30	32	756	776	771	755	829	743	746	798
81	26	66	32	33	761	783	777	762	842	756	757	812
82	26	67	33	34	765	786	783	767	854	770	771	827
83	25	67	35	35	770	793	790	773	865	785	784	832
84	25	66	36	36	774	798	796	780	877	799	801	835
85	25	67	37	37	778	809	803	788	889	811	812	839
86	25	68	38	38	776	814	809	790	894	816	817	836
87	25	74	39	39	783	820	810	801	904	819	819	830
88	25	82	40	40	785	825	818	803	908	816	816	823
89	25	93	41	41	796	835	823	813	919	814	815	819

	TLVY	TLNV	TLNK	TSAU	TAK1	TAK2	TAK3	TAK4	TAVT	TANT	TAVK	TANK
Index	before compressor T [°C]	after compressor T [°C]	after intercooler T [°C]	manifold temperature [°C]	exhaust manifold T1 [°C]	exhaust manifold T2 [°C]	exhaust manifold T3 [°C]	exhaust manifold T4 [°C]	before turbine T [°C]	after turbine T [°C]	before catalyst T [°C]	after catalyst T [°C]
90	24	41	22	25	732	726	723	706	729	611	614	675
91	25	50	20	26	751	768	758	733	747	655	655	705
92	26	58	24	27	765	776	770	753	770	687	690	737
93	26	63	26	29	773	787	780	765	799	712	713	762
94	25	67	31	31	775	794	787	772	820	733	733	789
95	25	63	32	31	775	792	788	773	836	750	750	804
96	25	68	30	32	783	801	795	782	852	763	763	818
97	25	67	31	33	787	806	799	787	866	777	777	832
98	25	67	33	35	788	808	804	791	875	790	791	842
99	25	67	33	36	790	810	809	795	884	801	803	845
100	25	66	37	37	794	813	813	800	892	812	815	848
101	25	68	38	38	796	821	819	804	901	821	820	847
102	25	68	40	39	793	823	824	809	907	827	827	844
103	25	72	41	40	800	833	829	818	917	832	831	846
104	25	77	42	42	803	843	837	822	925	836	837	846
105	25	83	44	43	807	842	834	827	927	832	833	839
106	25	95	44	44	801	839	829	819	918	815	816	817
107	25	101	45	45	797	835	820	809	910	801	792	802
108	24	41	19	25	763	763	755	738	767	676	677	718
109	25	49	22	27	784	796	786	768	782	712	714	748
110	25	58	24	28	797	804	797	783	807	731	733	777
111	25	65	27	29	801	814	807	795	832	749	750	796
112	25	69	30	32	803	815	808	797	847	758	759	810
113	25	68	31	33	805	818	810	803	866	776	777	830
114	25	67	33	34	806	822	812	808	878	790	791	843
115	25	68	35	36	808	825	817	810	886	800	801	837
116	25	66	36	37	807	824	817	811	891	809	808	840
117	25	66	39	38	807	825	820	819	900	820	820	843
118	25	67	40	40	807	825	823	819	904	823	824	841
119	25	67	42	41	807	832	826	822	911	830	832	845
120	25	72	43	42	812	837	834	827	919	835	838	846
121	25	75	48	45	812	840	835	831	927	836	839	843
122	25	82	47	46	821	852	845	839	933	840	839	848
123	25	92	48	47	826	857	852	841	937	835	836	838
124	25	106	48	48	821	843	843	827	922	813	815	811
125	25	47	18	25	800	799	789	776	812	720	721	754
126	25	54	23	27	811	819	809	793	819	743	742	780
127	25	64	25	28	829	831	822	812	845	763	764	805
128	25	70	27	30	833	839	827	824	866	776	777	825
129	25	67	30	32	831	841	828	824	880	791	792	840
130	25	68	32	33	829	839	826	822	888	801	801	841
131	25	68	33	34	822	830	817	813	887	801	803	825
132	25	65	35	36	812	822	812	810	885	803	803	819
133	25	68	37	37	813	821	815	817	891	808	809	823
134	25	69	39	39	819	825	818	824	900	817	817	828
135	25	70	39	39	819	830	823	827	908	825	826	834
136	25	69	43	42	820	834	831	831	915	835	837	840
137	25	73	43	43	824	836	836	834	919	835	835	838
138	25	80	46	45	826	848	840	837	924	835	834	836
139	25	86	48	47	829	847	840	843	926	831	831	832
140	25	99	48	48	836	855	844	852	931	826	828	826
141	25	107	50	50	832	848	829	839	918	809	810	808
142	24	42	18	26	817	827	815	795	838	746	744	792
143	25	51	22	27	831	842	832	813	856	775	775	820
144	25	58	24	28	841	851	840	828	879	794	795	838
145	25	62	27	30	848	860	848	837	894	809	809	848
146	25	65	29	31	849	862	847	841	906	820	821	850
147	25	67	31	33	843	857	840	835	906	822	821	842
148	25	68	33	35	834	848	835	829	904	820	821	836
149	25	68	34	36	832	844	835	831	906	824	824	837
150	25	70	37	37	829	842	834	831	907	826	827	835
151	25	71	38	38	832	847	843	836	916	835	836	841
152	25	72	40	40	829	844	836	834	915	836	837	838
153	25	69	43	43	826	844	842	834	918	842	843	842
154	25	78	44	46	825	851	846	841	921	838	839	836
155	25	82	49	48	823	849	841	847	919	834	837	830
156	24	100	51	51	815	843	835	842	908	808	813	805
157	25	64	19	25	862	853	843	843	891	780	781	789
158	25	69	23	27	871	873	865	859	899	810	811	830
159	25	68	26	29	869	868	858	859	905	815	815	833
160	25	68	29	31	864	862	849	857	907	819	820	833
161	25	66	32	34	857	856	844	851	912	828	828	839
162	25	65	34	35	855	855	844	851	917	835	835	845
163	25	67	36	37	852	857	844	852	921	841	840	848
164	25	67	40	40	845	847	839	846	916	839	841	842
165	25	68	42	42	841	844	837	845	915	839	838	839
166	25	70	43	43	839	848	838	851	920	844	843	842
167	25	71	46	45	840	848	842	848	921	845	845	841
168	25	77	49	48	841	851	841	849	921	842	843	836
169	25	85	51	51	835	844	833	846	911	827	826	820
170	25	100	52	52	822	836	818	837	894	799	799	794

	TOEL	TkWE	TkWA	PLVW	PLNV	PLNK	PSAU	O2VK	NATL	PAVT	PANT	PANK
Index	oil temperature [°C]	cooling water inlet T [°C]	cooling water outlet T [°C]	before compressor p [mbar]	after compressor p [mbar]	after intercooler p [mbar]	manifold pressure [mbar]	O2 before catalyst [%]	TC speed [1/min]	before turbine p [mbar]	after turbine p [mbar]	after catalyst p [mbar]
1	83	56	85	991	992	991	286	0.94	12777	1012	992	987
2	85	57	87	991	994	993	365	0.89	13342	1017	992	989
3	86	58	89	990	998	996	452	0.85	18187	1027	994	990
4	88	59	90	989	1005	1004	545	0.84	22371	1037	996	990
5	89	58	90	990	1016	1015	623	0.81	27935	1048	998	992
6	90	58	91	990	1026	1024	701	0.78	28738	1060	1002	994
7	91	60	91	991	1041	1038	793	0.74	33529	1073	1000	989
8	92	58	92	991	1060	1057	891	0.79	38897	1096	1006	996
9	92	58	92	991	1075	1072	966	0.74	43855	1109	1007	992
10	93	59	93	990	1095	1089	1077	0.71	48091	1132	1011	995
11	94	59	93	990	1099	1092	1091	0.36	48188	1134	1013	994
12	86	58	86	994	997	996	282	0.79	10234	1032	1003	999
13	89	58	88	991	999	998	360	0.73	15083	1043	1006	999
14	89	58	89	991	1009	1007	468	0.63	20452	1056	1007	1000
15	90	59	89	991	1023	1021	592	0.6	27350	1071	1009	999
16	91	59	90	991	1041	1038	672	0.59	33429	1092	1012	1000
17	92	59	91	990	1064	1061	756	0.6	40061	1118	1015	1000
18	93	59	91	990	1094	1091	822	0.58	47392	1149	1021	1001
19												
20	94	59	92	990	1130	1125	886	0.64	54536	1186	1025	1002
21	94	60	92	989	1168	1163	951	0.66	61073	1229	1029	1003
22	95	60	93	990	1203	1197	1128	0.61	67010	1272	1038	1006
23	98	61	94	989	1390	1375	1381	0.24	89284	1456	1048	1010
24	89	59	87	990	1005	1004	332	0.73	19704	1041	998	991
25	90	59	88	990	1016	1014	413	0.7	25483	1058	1001	991
26	92	59	89	989	1032	1030	497	0.68	31278	1080	1004	992
27	93	59	89	989	1059	1056	588	0.62	38956	1112	1008	992
28	94	60	90	989	1094	1090	674	0.61	47442	1150	1012	993
29	94	60	90	988	1130	1126	752	0.57	54544	1189	1017	994
30	96	60	91	988	1167	1162	816	0.58	61077	1228	1023	996
31	96	61	91	988	1206	1201	884	0.56	67497	1270	1028	997
32	97	61	92	987	1246	1240	952	0.58	73216	1317	1036	999
33	98	61	92	986	1281	1274	1016	0.59	78014	1353	1042	1001
34	98	61	92	986	1310	1301	1092	0.6	81022	1385	1048	1002
35	99	62	92	986	1320	1311	1172	0.59	82863	1404	1054	1004
36	99	62	93	984	1346	1335	1268	0.57	85615	1436	1062	1006
37	101	62	94	983	1519	1504	1500	0.3	102840	1625	1079	1012
38	101	62	94	981	1583	1568	1574	0.22	108339	1691	1082	1012
39	103	63	94	980	1800	1784	1791	0.11	124112	1904	1096	1017
40	92	59	88	991	1021	1019	333	0.72	26804	1075	1010	1001
41	93	59	88	991	1040	1037	410	0.66	33645	1103	1014	1001
42	94	60	89	991	1069	1066	494	0.66	41682	1139	1019	1001
43	96	60	90	990	1111	1107	579	0.63	51237	1185	1024	1002
44	97	61	91	989	1162	1157	664	0.66	60346	1237	1030	1004
45	97	61	91	989	1206	1200	739	0.62	67256	1281	1037	1005
46	98	61	91	989	1245	1239	801	0.65	72836	1320	1044	1006
47	99	62	91	988	1289	1281	875	0.63	78548	1368	1052	1008
48	99	62	91	987	1322	1314	930	0.61	82759	1405	1059	1011
49	100	62	92	986	1333	1322	994	0.62	83987	1432	1070	1014
50	101	62	92	985	1340	1327	1054	0.65	85240	1452	1078	1016
51	101	62	92	984	1345	1331	1138	0.62	86380	1464	1088	1019
52	102	63	92	983	1355	1340	1239	0.58	88308	1491	1099	1023
53	102	63	93	981	1417	1399	1352	0.54	95113	1564	1112	1026
54	103	63	93	980	1462	1442	1442	0.36	99711	1620	1121	1030
55	104	63	94	977	1710	1687	1685	0.2	119222	1872	1139	1036
56	94	61	88	991	1043	1040	338	0.64	33995	1102	1014	1000
57	95	60	89	990	1081	1078	435	0.64	44537	1151	1020	1001
58	97	61	89	990	1122	1117	510	0.65	53212	1198	1025	1001
59	98	61	90	989	1174	1169	592	0.56	62557	1252	1032	1003
60	99	62	90	989	1228	1222	680	0.6	70617	1309	1041	1004
61	100	62	90	988	1272	1264	753	0.58	76655	1358	1049	1006
62	100	62	91	987	1314	1305	815	0.61	82154	1408	1057	1008
63	101	63	91	986	1348	1338	886	0.6	86328	1452	1067	1010
64	102	63	91	984	1361	1349	949	0.62	88282	1481	1079	1013
65	103	63	91	984	1330	1315	1004	0.59	89825	1467	1089	1017
66	103	63	91	982	1339	1321	1093	0.54	87446	1487	1102	1021
67	104	63	92	980	1340	1320	1202	0.62	87627	1505	1115	1025
68	104	63	92	979	1340	1317	1269	0.49	88602	1523	1125	1028
69	105	63	92	978	1407	1383	1341	0.39	95560	1597	1133	1031
70	106	64	93	975	1478	1447	1448	0.29	101756	1682	1145	1036
71	106	64	93	974	1574	1542	1543	0.21	110637	1790	1155	1040
72	107	64	93	971	1688	1655	1656	0.14	119415	1918	1170	1045
73	96	61	88	991	1046	1043	310	0.67	36309	1110	1018	1000
74	98	61	89	990	1086	1082	395	0.68	46320	1161	1026	1002
75	99	62	89	989	1138	1132	475	0.69	56873	1221	1035	1003
76	100	62	90	988	1197	1190	556	0.62	66208	1285	1044	1005
77	101	62	90	988	1264	1256	649	0.65	75982	1363	1056	1007
78	103	62	91	986	1318	1308	732	0.68	82970	1426	1067	1009
79	103	63	90	985	1362	1350	808	0.66	87980	1485	1077	1012
80	104	63	91	984	1359	1345	877	0.61	88245	1495	1088	1015
81	105	63	91	983	1375	1359	950	0.58	90495	1530	1100	1018
82	106	64	91	981	1381	1361	1038	0.58	92267	1570	1114	1023
83	106	64	92	979	1388	1366	1103	0.5	92894	1593	1125	1026
84	107	64	92	977	1382	1356	1187	0.43	92944	1614	1139	1031
85	107	64	92	975	1394	1365	1266	0.36	95384	1654	1153	1036
86	108	64	92	973	1402	1370	1325	0.3	97130	1681	1163	1040
87	108	64	91	970	1475	1436	1433	0.21	104878	1788	1181	1046
88	108	65	92	968	1567	1525	1522	0.16	112481	1906	1196	1052
89	109	65	92	964	1705	1660	1656	0.13	122525	2077	1221	1061

	TOEL	TKWE	TKWA	PLVV	PLNV	PLNK	PSAU	O2VK	NATL	PAVT	PANT	PANK
Index	oil temperature [°C]	cooling water inlet T [°C]	cooling water outlet T [°C]	before compressor p [mbar]	after compressor p [mbar]	after intercooler p [mbar]	manifold pressure [mbar]	O2 before catalyst [%]	TC speed [1/min]	before turbine p [mbar]	after turbine p [mbar]	after catalyst p [mbar]
90	99	61	88	991	1106	1102	341	0.74	50252	1180	1027	1004
91	101	62	89	989	1176	1171	419	0.62	62812	1260	1036	1005
92	102	62	90	988	1257	1250	509	0.64	74568	1350	1048	1008
93	103	63	90	987	1304	1296	584	0.56	80935	1410	1060	1010
94	104	63	90	986	1362	1351	668	0.6	87677	1480	1073	1013
95	105	63	90	985	1331	1318	730	0.58	84706	1468	1084	1016
96	106	64	90	983	1394	1379	811	0.57	91748	1548	1099	1020
97	106	64	91	982	1390	1373	878	0.53	92478	1563	1112	1024
98	107	64	91	979	1397	1376	956	0.51	93877	1597	1128	1028
99	108	64	91	976	1391	1365	1037	0.43	94447	1637	1144	1034
100	109	65	91	975	1394	1356	1100	0.38	94686	1652	1156	1038
101	109	65	92	972	1398	1366	1191	0.3	97293	1705	1174	1045
102	110	65	92	970	1393	1355	1269	0.25	99037	1736	1191	1051
103	110	65	92	967	1449	1409	1333	0.2	103904	1822	1205	1056
104	110	65	92	963	1496	1447	1432	0.17	110171	1928	1231	1066
105	111	65	92	960	1564	1510	1507	0.14	115580	2035	1249	1073
106	112	65	92	955	1710	1653	1649	0.11	126513	2240	1276	1085
107	112	66	93	951	1796	1736	1732	0.09	132086	2363	1292	1092
108	102	62	88	990	1106	1102	350	0.69	50427	1187	1034	1005
109	104	63	89	989	1183	1177	438	0.63	64340	1280	1047	1007
110	105	63	89	988	1265	1257	519	0.61	76124	1378	1060	1010
111	106	63	90	986	1340	1329	599	0.55	85218	1464	1075	1013
112	107	64	90	984	1395	1381	676	0.56	91083	1534	1089	1017
113	108	64	90	982	1404	1388	761	0.54	93176	1569	1107	1022
114	109	64	90	980	1395	1375	823	0.51	93253	1584	1123	1026
115	109	65	91	978	1400	1377	893	0.37	94613	1615	1136	1031
116	110	65	91	977	1380	1355	946	0.33	93063	1619	1148	1035
117	111	65	91	973	1373	1340	1053	0.28	95459	1667	1171	1043
118	112	65	91	970	1385	1348	1128	0.24	98162	1712	1188	1049
119	112	65	92	967	1381	1339	1201	0.21	99827	1751	1205	1056
120	113	66	92	963	1426	1378	1296	0.18	104940	1846	1228	1065
121	113	66	92	960	1462	1408	1376	0.16	110514	1911	1258	1076
122	113	66	92	955	1539	1480	1480	0.14	116786	2074	1276	1084
123	114	66	92	949	1674	1611	1611	0.1	125677	2269	1305	1096
124	114	66	93	940	1838	1769	1769	0.07	137213	2524	1347	1115
125	106	63	88	989	1157	1151	359	0.6	59995	1250	1043	1006
126	107	63	89	988	1228	1221	431	0.63	71112	1336	1055	1009
127	108	64	90	986	1343	1332	530	0.53	85341	1470	1075	1013
128	109	64	90	984	1405	1392	605	0.51	92332	1500	1093	1017
129	110	64	90	982	1392	1376	665	0.49	91918	1558	1108	1021
130	111	65	90	980	1395	1375	727	0.38	93496	1588	1124	1026
131	111	65	91	977	1408	1383	812	0.25	95848	1642	1142	1033
132	112	65	91	974	1378	1349	877	0.19	94778	1649	1158	1038
133	113	65	91	971	1403	1370	960	0.17	98539	1710	1178	1046
134	114	66	91	967	1399	1358	1040	0.16	100861	1766	1198	1053
135	115	66	91	963	1397	1350	1123	0.15	102780	1818	1221	1062
136	115	66	91	959	1364	1308	1212	0.14	104281	1852	1247	1073
137	116	66	92	954	1408	1347	1285	0.12	109180	1950	1269	1082
138	116	67	92	950	1483	1417	1375	0.1	116563	2087	1295	1093
139	117	67	92	944	1541	1468	1467	0.09	121792	2216	1321	1105
140	117	67	92	935	1703	1624	1623	0.07	133517	2491	1367	1124
141	118	67	92	928	1800	1717	1716	0.06	139720	2664	1395	1138
142	110	64	89	989	1127	1120	367	0.63	55466	1233	1053	1008
143	111	64	89	987	1212	1201	454	0.57	69464	1339	1073	1012
144	112	64	90	984	1288	1274	525	0.49	80044	1437	1092	1016
145	112	65	90	983	1335	1319	579	0.42	86043	1498	1106	1020
146	113	65	90	980	1363	1343	652	0.34	89162	1563	1126	1026
147	114	65	91	977	1387	1362	730	0.25	93889	1631	1147	1033
148	114	66	91	973	1403	1373	814	0.2	97098	1697	1169	1042
149	116	66	91	969	1402	1366	888	0.17	99342	1743	1190	1050
150	116	67	91	965	1408	1366	966	0.15	102093	1802	1210	1058
151	117	67	91	961	1398	1347	1050	0.14	105105	1866	1237	1068
152	118	67	91	956	1387	1328	1126	0.12	106665	1918	1261	1079
153	119	67	92	952	1322	1253	1187	0.1	102805	1933	1286	1090
154	119	67	92	945	1419	1342	1288	0.09	116108	2116	1319	1104
155	120	67	92	940	1449	1365	1359	0.08	119241	2215	1342	1115
156	120	68	92	923	1670	1574	1571	0.06	136843	2649	1415	1150
157	113	64	89	987	1317	1308	397	0.35	82263	1419	1062	1008
158	114	65	90	984	1391	1379	485	0.4	90860	1524	1086	1013
159	115	65	90	982	1391	1375	552	0.29	91755	1557	1106	1019
160	116	66	90	979	1399	1380	614	0.23	93980	1602	1125	1024
161	117	66	90	976	1376	1350	686	0.21	93445	1637	1150	1033
162	117	66	90	972	1371	1340	755	0.19	95204	1679	1171	1041
163	119	67	91	968	1375	1336	840	0.16	98091	1747	1199	1051
164	119	67	91	964	1364	1318	915	0.15	100280	1795	1220	1060
165	120	67	91	959	1352	1296	989	0.13	103602	1866	1245	1070
166	121	67	91	954	1342	1277	1063	0.12	106447	1933	1275	1083
167	122	68	92	945	1312	1232	1165	0.1	109925	2035	1318	1102
168	122	68	92	941	1368	1282	1226	0.09	116227	2152	1339	1112
169	123	68	92	931	1430	1330	1320	0.08	123084	2359	1381	1133
170	123	68	92	917	1584	1475	1463	0.09	137980	2698	1436	1161

	PAVE	LAMS	PMAX	DPMX	APMX	BLBY	PRAIL	ESM	DRKW	WPED
Index	before exhaust pipe p [mbar]	lambda [-]	combustion maximum p [bar]	max. p rise dp/dt [bar*/KW]	max. p crank angle position [°KWnOTn]	blowby [l/min]	injection pressure [bar]	injection mass [mg]	throttle opening [%]	pedal position [%]
1	988	0.999	12.6	0.57	15.2	4.9	43	7.6	1.9	7
2	988	0.999	19.2	1.03	11.6	6.7	43	10.4	3.2	13
3	988	0.997	25.9	1.5	10.2	8.2	50	13.5	4.4	17
4	989	0.999	32.2	1.97	10	9.7	58	16.6	5.6	20
5	989	0.999	36.4	2.36	11.1	10.7	58	19.3	6.6	22
6	989	0.998	40.8	2.7	12.1	12	66	21.9	7.7	25
7	987	0.998	44.4	2.9	14.5	13.4	73	25.2	9.3	27
8	987	1.002	45	2.8	18	14.7	76	28.3	12	31
9	986	0.998	46.9	2.89	20.1	16.3	78	31.3	15.1	34
10	986	0.997	50.7	3.13	21.6	19.3	82	35.4	32.8	45
11	986	0.957	51.8	3.33	21.5	19.3	83	37.2	55.2	100
12	999	1.001	12.7	0.51	14.9	5.1	53	7.9	3.4	15
13	999	0.999	19.1	0.96	12.3	6.2	53	10.9	5.1	19
14	999	0.999	23.8	1.13	12.8	6.7	55	13.3	6.2	22
15	998	0.999	27	1.12	14.7	9.1	58	16	7.4	25
16	998	1	32.9	1.52	13.6	10.3	58	18.6	8.7	27
17	998	1	39.1	1.9	13.7	11.4	66	21.6	10.4	30
18	999	0.999	42.4	2.13	15.4	12.9	75	24.8	11.9	33
19										
20	999	0.999	45.3	2.32	16.6	14.4	79	28.2	13.5	35
21	999	0.999	46.2	2.39	19	16.3	85	31.8	15.1	38
22	999	0.998	51.2	2.61	22.7	19.1	93	38.3	24	44
23	999	0.958	53.2	2.55	29.5	22.8	93	48.9	84.5	98
24	990	1	14.7	0.66	15.7	5.7	65	8.3	5.4	17
25	990	1	18.9	0.78	13	6.9	72	10.9	7.1	21
26	990	1	24	1.07	12.3	7.9	85	13.3	8.4	24
27	991	0.999	29.9	1.43	11.6	9.5	93	16.3	10	26
28	991	0.999	35.5	1.77	11.4	11.2	93	19.2	11.5	29
29	991	0.999	40.4	2.08	11.8	12.5	93	21.9	12.9	31
30	991	0.999	44	2.31	12.7	13.3	95	24.7	14.2	33
31	991	0.999	46.2	2.39	14.6	14.7	97	27.8	15.5	36
32	992	0.998	50.6	2.71	15.7	15.9	100	31.3	17	40
33	992	0.998	54.3	2.96	16.8	17.7	103	34.5	18.9	42
34	992	0.998	59.3	3.45	16.8	19.3	103	37.4	20.5	44
35	992	0.998	60.5	3.41	18.6	20	103	39.9	23.6	46
36	992	0.998	61.3	3.19	20.7	22	103	43.3	29.3	48
37	993	0.976	62.8	2.88	26	24.7	103	52.5	51.2	52
38	993	0.96	65.2	3.09	27	25.8	103	55.5	71.1	56
39	994	0.921	65.6	2.96	30.6	28.6	103	65.4	99.8	100
40	1000	1.001	15.1	0.7	15.8	6.9	76	8.7	6.8	20
41	1000	1.001	20.2	0.91	12.9	8.5	84	11.3	8.6	24
42	1000	0.999	26.2	1.3	11.5	9.7	91	13.9	10.1	26
43	1000	1	31.9	1.68	11	10.8	93	16.7	11.5	29
44	1000	1	38	2.09	10.8	12.1	93	19.7	13	32
45	1000	1	42.8	2.41	11.2	13.2	94	22.5	14.2	35
46	1000	1	48.2	2.82	11	14.4	96	25	15.3	37
47	1000	1.001	53	3.23	11.6	16	98	28.1	16.6	39
48	1000	1	56	3.38	12.8	17.3	100	30.9	17.7	41
49	1000	1	57.3	3.3	15.8	18.7	103	34.3	19.7	44
50	1001	1	63.3	3.85	14.5	19.9	103	36.6	21.4	46
51	1001	1	65.2	3.82	16.1	20.9	103	39.4	24.2	48
52	1001	1	66.7	3.8	17.6	22.4	103	42.8	29.4	51
53	1001	1	66.8	3.54	20.6	23.6	103	46.6	36.5	54
54	1002	0.983	67.9	3.46	22.4	24.5	103	50.3	63.2	55
55	1002	0.941	70.7	3.42	25.9	28.7	104	59.7	99.8	100
56	999	1.002	13.5	0.45	16.8	7.3	93	9.1	8.3	21
57	999	1.001	21.5	0.91	13.9	8.1	93	12.8	10.3	25
58	999	1.001	24.6	1.06	14.1	9.2	93	14.4	11.6	28
59	999	1	30.3	1.42	13.4	10.3	93	17.1	13	31
60	999	1	36.3	1.82	12.9	12.3	93	20.2	14.5	34
61	1000	1	41.7	2.19	12.5	14.3	94	23	15.7	36
62	1000	1	47	2.57	12.1	14.6	97	25.5	16.8	39
63	1000	1	52	2.91	12.6	15.3	99	28.4	18.1	41
64	1000	1	56.5	3.26	12.8	16.3	102	31.6	19.6	43
65	1001	1	60.6	3.66	12.8	16.5	103	34.3	22.6	45
66	1001	0.999	64.7	3.85	13.9	18.4	104	37.6	25.2	48
67	1002	1	68.1	3.92	14.7	19.8	104	40.4	30.6	50
68	1002	0.989	72.2	4.18	14.9	23.1	104	43.4	39.9	53
69	1002	0.976	73.7	4.21	16.6	24.1	104	46.5	41.9	59
70	1003	0.955	76.2	4.41	18.2	25.5	104	50.9	97.6	62
71	1003	0.932	76.9	4.26	20.3	26.4	104	55.5	97.5	67
72	1004	0.905	76.7	3.95	22.7	26.7	104	61.2	97.7	100
73	999	1	16.1	0.77	13.3	4.2	97	9.2	9.6	22
74	999	1	21.9	1.13	12.4	5.2	103	12.1	11.7	26
75	1000	1	27	1.41	12.3	6.8	103	14.8	13.3	29
76	1000	1	31.5	1.62	12.9	10.1	103	17.7	14.7	32
77	1000	1	37.4	2	12.6	12	103	20.9	16.2	35
78	1000	1	43.4	2.42	11.7	13.8	103	23.9	17.4	38
79	1000	1	49.5	2.91	10.9	15.3	103	26.4	18.5	41
80	1001	1	54	3.18	11.4	15.8	103	29.1	20.2	42
81	1001	1	58.7	3.45	11.8	16.2	103	31.8	21.7	44
82	1002	1	64.5	3.87	11.8	16.8	104	35.1	24.4	47
83	1002	0.993	67.4	4.01	12.7	17.4	104	37.7	26.3	49
84	1003	0.985	71.2	4.19	13.7	18.9	104	41	30.4	52
85	1003	0.976	74.6	4.34	14.6	19.8	104	44.6	36.1	54
86	1004	0.963	77.2	4.56	15.5	21.1	104	47.4	45.3	57
87	1004	0.936	77.4	4.36	18.4	22.3	104	52.4	99.8	63
88	1005	0.912	80.6	4.45	19.5	22.9	104	57.5	99.8	68
89	1006	0.883	82.8	4.37	21.5	27	105	64.6	99.8	100

	PAVE	LAMS	PMAX	DPMX	APMX	BLBY	PRAIL	ESM	DRKW	WPED
Index	before exhaust pipe p [mbar]	lambda [-]	combustion maximum p [bar]	max. p rise dp/dt [bar°K/W]	max. p crank angle position [°Kw/OTh]	blowby [l/min]	injection pressure [bar]	injection mass [mg]	throttle opening [%]	pedal position [%]
90	1002	1	18.8	0.97	11.2	8.6	100	10	11.1	24
91	1002	0.999	24.5	1.34	10.5	9.8	103	12.7	12.7	27
92	1003	0.999	30.1	1.67	10.9	11.3	103	15.8	14.3	31
93	1003	0.999	35	1.95	11.1	12.8	103	18.6	15.8	33
94	1003	0.999	40.3	2.28	11.1	14	103	21.4	17.1	36
95	1004	0.999	44.4	2.57	11.3	15.5	103	23.7	18.9	38
96	1004	0.999	50.6	3.01	11	16.1	103	26.9	20.1	40
97	1004	0.999	54.6	3.26	11.5	16.6	104	29.4	21.6	42
98	1005	0.999	60.5	3.65	11.3	17.7	104	32.3	23.4	45
99	1006	0.991	66.4	4.08	11.3	18.7	104	35.5	26.4	47
100	1006	0.985	69.1	4.17	12.4	18.7	104	38.2	28.7	49
101	1007	0.969	74.8	4.59	12.8	19.4	104	42.3	32.8	52
102	1008	0.954	80	4.88	13.1	19.8	104	46.1	40.9	54
103	1008	0.937	77.9	4.48	16	20.1	104	49.2	43	57
104	1009	0.919	80.4	4.49	17.9	22.8	105	54.8	63.7	59
105	1010	0.887	81.8	4.49	19.3	24.3	105	60.1	99.8	64
106	1012	0.835	84.2	4.47	21.2	26.7	105	69.2	99.8	72
107	1013	0.808	88	4.65	21.3	27.8	105	75	99.9	100
108	1002	1.001	18.6	0.9	12.6	15.8	101	10.4	12.4	24
109	1003	1	24.6	1.29	12	15.6	103	13.3	14.3	28
110	1003	1	29.9	1.61	12.2	12.6	103	16.3	15.8	31
111	1003	0.999	35.4	1.94	12	13.7	103	19.2	17.2	34
112	1004	0.999	41.5	2.37	11	14.7	103	21.9	18.5	37
113	1004	1	47.4	2.74	11.1	15.7	103	25.1	20.4	40
114	1005	0.995	52.1	3.11	11.2	16.8	104	27.7	22	41
115	1005	0.98	57.1	3.49	11.3	17.8	104	30.8	23.6	43
116	1006	0.974	61.7	3.85	11	18.6	104	33.1	25.3	44
117	1007	0.963	69.1	4.35	11.4	19.4	104	37.5	29.4	48
118	1008	0.951	74	4.67	11.7	20.5	104	40.5	32.2	50
119	1008	0.943	76.7	4.74	12.9	21.1	104	43.6	37.4	52
120	1009	0.928	81.8	5.16	13.6	21.4	105	48.1	44.1	56
121	1011	0.902	82.5	4.97	15.6	22.9	105	53	54.9	58
122	1011	0.886	81.3	4.54	18.3	24.1	105	57.3	89.2	62
123	1013	0.861	81.7	4.28	20.5	26.4	105	63.6	99.9	69
124	1015	0.792	79.4	3.81	23.8	28.2	106	75.4	99.9	100
125	1002	1.001	20.2	0.98	12.7	20.8	103	11.2	13.7	25
126	1002	1	25.6	1.33	11.7	22	103	13.7	15.3	28
127	1003	0.999	32.7	1.8	11.4	23.1	103	17.3	17.1	32
128	1003	0.999	38	2.12	11.4	16.6	103	20.3	18.6	36
129	1004	0.999	42.4	2.41	11.4	15.4	103	22.5	20.2	38
130	1004	0.983	47.7	2.79	11.3	16.1	104	25.5	21.7	39
131	1005	0.949	55.6	3.45	10.6	16.8	104	29.7	23.7	42
132	1006	0.928	61	3.88	10.6	17.3	104	33.3	25.8	44
133	1007	0.919	67.8	4.39	10.2	18	104	36.7	27.6	46
134	1007	0.914	72.6	4.65	10.7	18.1	104	39.8	30.7	49
135	1009	0.91	78.5	5.09	10.8	19.2	105	43.6	34.3	52
136	1010	0.899	79.6	4.98	13.3	19.6	105	47.6	45.1	54
137	1011	0.885	84.3	5.23	13.6	21.7	105	51.3	50.1	56
138	1012	0.863	84.2	4.94	15.9	22.8	105	55.9	55.2	58
139	1014	0.846	89.3	5.22	16.1	24.3	105	60.6	99.8	63
140	1016	0.818	90.7	4.97	18.2	26	106	68.8	99.9	70
141	1018	0.785	89.2	4.58	20.3	28.2	106	75	99.9	100
142	1002	1.001	21	1.02	13.1	27.7	103	11.7	15.5	27
143	1003	1.001	27.8	1.46	12.2	27.1	103	15.1	17.7	31
144	1003	0.999	32.7	1.72	12.1	26.9	103	17.9	19.1	35
145	1004	0.994	36.3	1.95	12.3	26.2	103	19.9	20.1	37
146	1004	0.979	42.3	2.35	12.1	19.8	103	23.3	21.8	39
147	1005	0.953	50.1	2.96	11.3	18	103	27.2	23.5	42
148	1006	0.929	57.9	3.6	10.6	17.6	104	31.3	25.5	44
149	1007	0.917	63.7	4.02	10.5	18.6	104	34.6	27.5	46
150	1008	0.905	70	4.48	10.2	18.9	104	38.1	29.6	49
151	1009	0.896	74.1	4.63	11.3	19.6	104	42.1	33.3	52
152	1011	0.879	81	5.23	10.9	21.3	105	46	37.9	55
153	1012	0.865	80.8	4.91	13.6	21.7	105	50	51.8	56
154	1014	0.848	85.6	5.11	14.2	22.6	105	55.2	55	60
155	1015	0.823	86.8	5.07	15.4	23.9	106	59.6	76.9	62
156	1020	0.762	90.3	4.82	18	25.3	106	74	99.9	100
157	1000	0.98	23.6	1.19	12.2	35.7	103	12.9	15.7	29
158	1001	0.984	29.3	1.5	12.4	33.7	103	16.2	17.9	34
159	1002	0.964	35	1.87	12.2	34.1	103	19.7	20	37
160	1002	0.942	39.7	2.17	12.1	33.6	103	22.5	21.6	40
161	1003	0.927	46.2	2.61	11.6	33.3	104	26.1	23.9	42
162	1004	0.919	51.8	3.04	11.4	26.3	104	29.2	25.8	44
163	1005	0.904	57.4	3.4	12	22.9	104	33.1	28.1	47
164	1007	0.888	65	4.02	10.8	21	104	36.8	30.6	49
165	1008	0.872	71.4	4.54	10.6	20.7	105	40.5	34.3	52
166	1009	0.859	75.6	4.7	11.6	21.3	105	44.8	38.9	55
167	1011	0.839	79.7	4.72	13.6	21.6	105	50.5	54.1	59
168	1013	0.823	82.4	4.85	13.9	22.5	106	53.5	56.8	61
169	1015	0.789	89.8	5.27	14	24.5	106	61.2	99.9	68
170	1019	0.738	90.4	4.83	16.7	26.6	107	72.6	99.9	100





	Engine Speed (cycle average), Part R4	Brake Torque, Part R4	Brake Power (kW), Part R4	BMEP - Brake Mean Effective Pressure, Part R4	Angle at End of Injection, Part in-1	Angle at Start of Injection, Part in-1	Valve-Open Timing Angle, Part Intake Valve 1 nk Angle (4-str	Fuel Flow, Part R4	BSFC - Brake Specific Fuel Consumption, Part R4	Combustion Start, Part Cyl1	Average Inlet Temperature; Part Turbo- Comp
Index	RFM	N·m	kW	bar	deg	deg	deg	kg/hr	g/kW·h	deg	K
96	4000	15.7898	6.61401	0.999955	508.476	381.237	363.359	5.03684	761.844	-31.086	294.439
97	4000	31.582	13.229	2.00006	509.575	382.086	365.064	6.34943	479.962	-29.2112	294.353
98	4000	47.3701	19.8424	2.99991	509.32	381.918	364.735	7.67652	396.875	-27.7746	294.263
99	4000	63.1657	26.4588	4.00024	509.526	382.054	364.48	9.01267	340.63	-26.2761	294.167
100	4000	78.9404	33.0665	4.99923	508.19	380.956	364.823	10.35	313.006	-24.768	294.063
101	4000	94.7482	39.6881	6.00033	511.518	384.231	365.122	11.6725	294.107	-23.4163	293.951
102	4000	110.538	46.3022	7.00031	509.372	381.953	364.799	12.9708	280.132	-22.0759	293.829
103	4000	126.323	52.9142	7.99996	508.529	381.172	364.223	14.2889	270.039	-20.5326	293.695
104	4000	142.125	59.5332	9.00067	513.721	386.079	363.933	15.6688	263.195	-19.5339	293.551
105	4000	157.908	66.1443	10.0002	518.411	390.807	363.898	17.1727	259.625	-19.2781	293.399
106	4000	173.696	72.7578	11.0001	522.561	395.263	363.91	18.9732	260.772	-18.1028	293.221
107	4000	189.472	79.3658	11.9991	526.828	399.446	364.079	20.961	264.106	-16.3707	293.022
108	4000	205.275	85.9856	12.9999	528.854	401.28	364.226	23.1248	268.938	-14.8522	292.786
109	4000	221.044	92.5908	13.9986	529.177	401.661	363.867	25.3342	273.614	-13.2057	292.518
110	4000	236.857	99.2143	15	529.558	402.001	364.107	27.8848	281.056	-11.0126	292.187
111	4000	252.648	105.829	16	530.193	403.025	362.816	30.7129	290.213	-8.58235	291.766
112	4000	268.435	112.442	16.9998	530.779	403.249	361.125	33.7078	299.78	-6.73941	291.315
113	4000	284.226	119.056	17.9998	522.535	395.456	359.819	36.5242	306.781	-6.03479	290.867
114	4500	15.7917	7.44167	1.00008	537.221	381.951	365.088	5.94333	788.656	-27.9447	294.388
115	4500	31.5828	14.883	2.00011	537.042	382.009	365.008	7.39787	497.068	-26.0387	294.287
116	4500	47.3686	22.3304	3.00096	538.656	382.969	364.992	8.89991	398.556	-25.2181	294.176
117	4500	63.1715	29.7689	4.0006	537.702	382.081	364.998	10.4239	350.16	-24.7181	294.054
118	4500	78.9621	37.21	5.00061	537.08	381.926	364.747	11.9163	320.246	-23.6616	293.926
119	4500	94.7572	44.6533	6.0009	536.961	381.27	364.574	13.3977	300.038	-22.4486	293.788
120	4500	110.544	52.0928	7.00069	536.776	381.724	364.736	14.8999	286.025	-21.0061	293.634
121	4500	126.332	59.5327	8.00053	539.365	384.001	365.002	16.4374	273.817	-20.2683	293.466
122	4500	142.123	66.9739	9.00054	543.712	388.137	365.088	18.0636	269.712	-19.7685	293.284
123	4500	157.892	74.4049	9.99918	547.39	392.019	364.948	19.8757	267.129	-19.1821	293.083
124	4500	173.696	81.8521	11	549.122	393.667	364.704	21.8724	267.219	-18.4035	292.861
125	4500	189.476	89.2887	11.9994	547.81	392.763	364.468	24.1548	270.524	-16.6162	292.697
126	4500	205.291	96.7411	13.0009	548.605	393.227	364.148	26.5942	274.901	-14.5445	292.599
127	4500	221.066	104.175	13.9999	554.346	398.958	363.161	29.4114	282.327	-12.3164	291.932
128	4500	236.857	111.616	15	548.062	392.842	361.696	32.6053	292.119	-9.73043	291.499
129	4500	252.638	119.053	15.9993	541.939	386.525	361.114	36.3709	305.503	-6.81164	291.876
130	4500	268.429	126.494	16.9994	524.088	369.012	360.004	40.5396	320.486	-4.03664	290.168
131	5000	15.8049	8.27541	1.00091	565.307	382.012	364.903	6.93203	798.656	-27.0838	294.337
132	5000	31.593	16.542	2.00076	565.022	381.75	365.174	8.50381	514.072	-25.6545	294.22
133	5000	47.3822	24.8093	3.00068	562.547	379.294	365.113	10.0905	406.723	-24.5271	294.09
134	5000	63.1764	33.0791	4.00091	564.296	381.036	364.96	11.7233	354.403	-23.6549	293.946
135	5000	78.9523	41.3393	4.99999	568.862	385.613	364.873	13.4085	324.353	-22.8719	293.787
136	5000	94.733	49.6021	5.99937	564.839	381.602	364.787	15.0776	303.927	-21.7405	293.622
137	5000	110.555	57.8867	7.00139	563.859	380.683	364.629	16.9143	292.197	-20.5786	293.428
138	5000	126.309	66.1353	7.99905	565.01	381.913	364.408	18.7265	283.155	-20.6092	293.224
139	5000	142.113	74.4103	8.99991	564.947	381.891	364.177	20.6928	278.091	-20.8174	292.985
140	5000	157.912	82.6827	10.0005	568.578	385.544	364.005	22.7236	274.829	-20.058	292.725
141	5000	173.698	90.9429	10.9995	562.396	378.743	363.961	24.9657	274.521	-19.0917	292.435
142	5000	189.489	98.2165	12.0002	547.023	363.423	364.064	27.5664	277.841	-17.1842	292.096
143	5000	205.275	107.482	12.9999	548.502	364.937	364.202	30.8069	286.625	-13.9369	291.855
144	5000	221.069	115.752	14.0001	548.173	364.629	364.018	34.4376	297.513	-12.2726	291.146
145	5000	236.858	124.019	15.0001	546.494	362.962	362.813	38.5022	310.455	-10.9519	290.544
146	5000	252.662	132.294	16.0009	546.047	362.514	361.132	43.2926	327.246	-8.54358	289.766
147	5000	268.439	140.554	17	547.383	363.848	360.593	48.8748	347.729	-5.1955	288.678
148	5500	15.7879	9.09319	0.999837	559.307	373.907	364.167	8.36201	919.691	-27.9859	294.256
149	5500	31.583	18.1905	2.00012	559.645	374.264	364.427	10.007	550.126	-26.5778	294.122
150	5500	47.3722	27.2844	3.00004	557.844	372.433	364.473	11.6774	427.988	-23.9241	293.972
151	5500	63.1599	36.3775	3.99987	553.309	368.004	364.273	13.4894	370.816	-22.9849	293.796
152	5500	78.9665	45.4814	5.00089	546.846	361.531	363.935	15.4033	338.673	-22.4108	293.6
153	5500	94.7281	54.5595	5.99905	550.011	364.82	363.587	17.4142	319.179	-21.5992	293.387
154	5500	110.519	63.6544	6.99908	551.023	365.906	363.307	19.6206	308.237	-20.9824	293.141
155	5500	126.312	72.7504	7.99923	546.996	361.936	363.115	21.9735	302.04	-20.9982	292.855
156	5500	142.104	81.8458	8.99931	546.298	360.471	363.001	24.2004	295.883	-20.4747	292.565
157	5500	157.906	90.9471	10	545.77	360	362.955	26.5052	291.436	-19.0128	292.236
158	5500	173.694	100.035	10.9993	545.672	360	362.973	29.0314	290.213	-17.3258	291.865
159	5500	189.469	109.126	11.9989	545.986	360.389	363.036	32.0568	293.76	-16.3343	291.427
160	5500	205.276	118.231	13	545.693	360.116	363.056	35.8063	302.851	-14.9729	290.895
161	5500	221.056	127.319	13.9993	545.564	360	362.792	40.4845	317.977	-11.9863	290.208
162	5500	236.845	136.413	14.9992	545.596	360	361.761	45.8765	336.306	-10.0992	289.375
163	5500	252.636	145.508	15.9993	546.082	360.423	358.977	51.3238	352.721	-8.85915	288.465
164	6000	15.7928	9.92291	1.00015	568.782	372.198	361.039	9.8677	994.435	-27.8503	294.164
165	6000	31.581	19.8429	2	569.438	372.909	360.055	11.7096	590.117	-25.191	294.007
166	6000	47.3692	29.763	2.99986	562.008	366.183	359.839	13.5841	456.41	-23.4387	293.829
167	6000	63.1569	39.6827	3.99968	556.498	360	360.253	15.5852	392.998	-22.5212	293.622
168	6000	78.9454	49.6028	4.99955	556.63	360.17	360.838	17.7704	358.253	-22.0226	293.385
169	6000	94.7387	59.5261	5.99973	556.098	360	361.324	20.1136	337.896	-21.4269	293.118
170	6000	110.529	69.4475	6.99973	556.73	360	361.673	22.6514	326.165	-20.7262	292.816
171	6000	126.315	79.3662	7.99945	556.636	360.161	361.919	25.393	319.948	-19.9848	292.474
172	6000	142.122	89.298	9.00049	556.415	360	362.087	29.4766	318.893	-19.3693	292.057
173	6000	157.915	99.2212	10.0007	556.359	360.211	362.185	31.4696	317.168	-18.5413	291.824
174	6000	173.703	109.141	11.0004	556.898	360.125	362.202	34.6222	317.226	-16.8702	291.14
175	6000	189.475	119.051	11.9993	556.621	360	362.102	38.3265	321.935	-14.6371	290.557
176	6000	205.265	128.972	12.9993	556.849	360.183	361.784	43.0485	333.781	-12.9635	289.811
177	6000	221.056	138.893	13.9993	557.757	360.931	361.024	48.7881	351.263	-11.935	288.88



	Average Outlet Temperature; Part Turbo- Comp	Mass Averaged Temperature (Outlet), Part After Cooler	Mass Averaged Temperature; Part SR-TV- 23	Mass Averaged Temperature (Outlet), Part Exh_Manifold- 1	Mass Averaged Temperature (Outlet), Part Exh_Manifold- 2	Mass Averaged Temperature (Outlet), Part Exh_Manifold- 3	Mass Averaged Temperature (Outlet), Part Exh_Manifold- 4	Average Inlet Temperature; Part Turbo- Turbine	Average Outlet Temperature; Part Turbo- Turbine	Mass Averaged Temperature; Part Cat-IN	Mass Averaged Temperature; Part Cat-Out
Index	K	K	K	K	K	K	K	K	K	K	K
96	308.448	298.814	300.961	1144.55	1164.5	1159.79	1145.83	1118.92	1104.9	1090.82	1087.99
97	316.08	299.166	300.169	1157.54	1171.92	1166.84	1157.33	1133.43	1115.18	1102.08	1099.38
98	324.63	299.527	300.08	1166.65	1178.78	1173.53	1166.31	1146.56	1123.1	1111.26	1108.67
99	332.866	300.174	300.416	1175	1187.36	1182.32	1175.02	1158.94	1130.29	1119.84	1117.32
100	339.748	300.801	300.841	1184.01	1197.08	1192.5	1183.88	1171.39	1138	1128.56	1126.11
101	341.858	301.153	301.106	1192.11	1204.57	1200.76	1192.07	1181.78	1145.37	1136.34	1135.95
102	339.941	301.219	301.144	1200.27	1210.91	1207.97	1200.79	1191.59	1153.94	1149.99	1147.68
103	337.833	301.244	301.166	1207.58	1218.83	1216.66	1208.36	1201.51	1162.94	1161.94	1159.7
104	337.49	301.384	301.3	1213.39	1228.66	1227.63	1213.78	1211.21	1171.15	1172.43	1170.29
105	336.601	301.463	301.379	1207.19	1225.07	1226.72	1207.61	1209.32	1168.46	1171.93	1169.92
106	340.222	301.937	301.808	1201.67	1219.19	1221.39	1202.2	1205.07	1161.09	1166.31	1164.43
107	338.211	301.923	301.799	1196.46	1213.35	1216.39	1197.2	1202.29	1158.13	1165.53	1163.77
108	343.757	302.614	302.423	1194.89	1211.59	1215.19	1195.55	1201.06	1152.62	1161.67	1160.02
109	341.554	302.586	302.414	1193.86	1210.08	1214.6	1194.58	1202.05	1153.4	1164.36	1162.83
110	349.64	303.601	303.338	1200.09	1216.63	1221.58	1200.64	1207.7	1152.94	1165.68	1164.22
111	354.05	304.314	303.998	1207.72	1224.03	1229.39	1207.9	1215.23	1156.1	1170.88	1169.51
112	363.942	305.633	305.214	1212.75	1229.31	1234.99	1212.49	1219.35	1152.73	1169.55	1168.25
113	372.921	306.872	306.363	1210.32	1226.83	1233.12	1209.85	1216.49	1143.33	1161.79	1160.55
114	313.261	299.897	301.176	1151.51	1192.33	1188.58	1152.78	1138.15	1121.16	1107.3	1104.45
115	322.56	300.252	300.875	1169.21	1193.55	1189.54	1169	1151.23	1129	1116.4	1113.82
116	332.27	300.963	301.17	1181.69	1195.36	1191.19	1182.46	1163.5	1135.26	1124.28	1121.73
117	340.556	301.707	301.666	1191.16	1199.62	1195.56	1192.18	1174.59	1140.79	1131.15	1128.68
118	342.01	302.039	301.906	1197.21	1205.57	1202	1198.44	1184.36	1147.58	1140.84	1139.44
119	342.14	302.267	302.093	1203.2	1213.87	1210.97	1204.35	1194.34	1155.29	1151.58	1149.26
120	343.1	302.55	302.34	1209.66	1223.46	1221.1	1210.84	1204.5	1162.99	1161.97	1159.74
121	343.181	302.757	302.535	1213.18	1232.73	1231.08	1214.28	1212.94	1169.55	1171.14	1169.02
122	342.718	302.908	302.683	1212.09	1233.99	1234.9	1213.18	1215.68	1170.94	1175.01	1173.01
123	342.767	303.095	302.861	1206.82	1226.26	1228.03	1207.71	1211.41	1165.31	1171.67	1169.79
124	339.733	302.992	302.791	1198.38	1215.59	1218.32	1199.27	1204.78	1159.04	1167.75	1166.01
125	344.64	303.642	303.389	1194.78	1211.26	1214.52	1195.58	1201.7	1152.01	1162.48	1160.83
126	341.249	303.497	303.288	1190.89	1205.63	1209.95	1191.62	1199.59	1150.42	1162.96	1161.43
127	348.076	304.398	304.123	1190.95	1205.25	1209.98	1191.46	1199.5	1144.89	1159.34	1157.88
128	348.904	304.724	304.454	1192.67	1205.57	1210.99	1192.36	1201.66	1144.81	1161.37	1160
129	359.681	306.193	305.816	1198.04	1211.73	1217.49	1197.68	1206.79	1141.39	1160.45	1159.14
130	371.487	307.873	307.375	1204.58	1219.02	1225.09	1204.2	1212.99	1138.37	1159.91	1158.65
131	318.765	300.981	301.996	1171.24	1184.26	1181.69	1172.47	1145.29	1125.23	1112.31	1109.57
132	329.395	301.559	302.053	1191.87	1205.17	1202.34	1191.66	1168.85	1142.38	1130.47	1127.87
133	337.921	302.31	302.469	1208.94	1222.76	1219.82	1208.56	1190.67	1158.15	1147.64	1145.1
134	341.759	302.821	302.79	1218.03	1231	1226.8	1218.89	1203.77	1166.98	1159.43	1156.96
135	344.495	303.27	303.114	1221.81	1235.32	1231.88	1223.17	1211.77	1171.37	1166.91	1164.53
136	338.504	303.031	302.877	1224.4	1237.89	1236.56	1224.72	1218.49	1176.63	1176.65	1176.36
137	342.194	303.539	303.32	1224.8	1238.15	1237.68	1225.11	1221.33	1177.81	1180.21	1178.04
138	335.649	303.167	302.981	1218.3	1231.18	1231.38	1218.72	1218.62	1176.93	1182.82	1180.77
139	339.316	303.677	303.436	1214.81	1227.5	1228.34	1215.1	1216.94	1171.87	1179.77	1177.82
140	338.89	303.818	303.588	1211.4	1223.14	1224.83	1211.53	1215.48	1169.26	1179.34	1177.48
141	338.529	303.96	303.708	1205.55	1217.38	1219.91	1205.72	1212.1	1164.83	1176.91	1175.13
142	337.773	304.066	303.816	1201.02	1210.57	1213.85	1201.08	1208.77	1160.6	1174.62	1172.91
143	343.186	304.842	304.555	1200.95	1210.23	1213.95	1200.88	1209.26	1156.28	1172.45	1170.81
144	348.567	305.649	305.335	1192.7	1201.26	1205.44	1192.4	1201.64	1143.81	1162.21	1160.64
145	355.57	306.678	306.317	1181.89	1189.46	1193.96	1181.22	1191	1127.17	1147.99	1146.48
146	367.938	308.425	307.947	1178.57	1186.07	1190.7	1177.82	1187.23	1113.84	1137.29	1135.83
147	383.023	310.847	310.028	1184.97	1192.71	1197.44	1184.14	1192.96	1107.95	1134.17	1132.75
148	318.545	300.619	301.844	1186.78	1182.52	1180.27	1188.61	1155.65	1134.17	1123.72	1121.1
149	326.192	301.249	301.698	1206.68	1202.99	1200.3	1207.85	1179.07	1151.89	1143.47	1140.93
150	330.308	301.724	301.767	1223.01	1229.19	1216.29	1224.01	1199.61	1167.98	1162.59	1160.11
151	335.982	302.356	302.176	1233.32	1230.38	1228.14	1234.33	1214.32	1177.68	1174.99	1172.58
152	338.225	302.758	302.478	1235.94	1233.88	1232.06	1236.88	1221.16	1181.21	1181.82	1179.51
153	338.768	303.021	302.726	1231.73	1231.26	1230.08	1232.71	1221.38	1179.23	1183.09	1180.89
154	342.954	303.606	303.265	1226.32	1226.86	1226.4	1227.42	1219.21	1173.18	1179.57	1177.47
155	347.093	304.222	303.86	1220.31	1222.11	1222.05	1221.47	1216	1165.87	1174.67	1172.68
156	339.893	303.747	303.48	1211.45	1214.55	1215.26	1212.69	1211.73	1164.03	1175.92	1173.99
157	339.531	303.904	303.661	1207.77	1211.42	1212.71	1209.12	1210.66	1161.73	1175.73	1173.87
158	339.485	304.094	303.884	1203.11	1207.28	1209.06	1204.51	1208.36	1158.05	1174.02	1172.21
159	340.492	304.4	304.23	1192.67	1196.88	1199.07	1193.85	1199.86	1147.61	1165.55	1163.81
160	345.207	305.128	304.963	1178.64	1182.73	1185.05	1179.48	1186.93	1130.29	1150.42	1148.75
161	353.588	306.324	306.122	1187.11	1170.73	1173.04	1167.59	1175.62	1111.91	1134.64	1133.04
162	365.681	308.048	307.745	1152.26	1155.21	1157.57	1152.55	1160.33	1087.41	1112.68	1111.15
163	378.061	309.856	309.468	1131.85	1134.06	1136.26	1132.05	1139.42	1057.52	1084.79	1083.35
164	335.286	301.724	303.213	1201.12	1196.24	1191.34	1200.75	1169.17	1139.26	1126.49	1125.99
165	342.871	302.505	303.167	1217.64	1211.23	1206.26	1218.8	1188.42	1152.65	1144.1	1141.68
166	343.534	302.858	302.964	1228.36	1221.61	1216.75	1227.59	1203.74	1164.63	1159.9	1157.54
167	342.07	303.034	302.863	1232.98	1232.98	1228.58	1221.8	1232.71	1213.35	1171.86	1169.56
168	340.076	303.131	302.883	1231.61	1226.1	1221.98	1231.62	1216.9	1174.2	1178.16	1175.95
169	338.364	303.217	302.954	1225.55	1221.22	1218.03	1226.09	1215.62	1171.77	1179.33	1177.2
170	337.065	303.315	303.082	1216.83	1213.05	1210.7	1217.56	1210.89	1165.98	1176.63	1174.6
171	336.433	303.463	303.264	1206.32	1203.53	1202.03	1207.53	1204.25	1158.1	1171.42	1169.47
172	345.852	304.64	304.345	1200.35	1187.72	1186.55	1201.46	1189.16	1154.74	1161.29	1159.42
173	342.72	304.548	304.337	1189.47	1187.1	1186.74	1190.84	1191.51	1138.39	1156.31	1154.51
174	341.505	304.641	304.521	1180.21	1178.11	1178.37	1181.89	1184.85	1130.95	1150.92	1149.17
175	343.8	305.129	305.096	1170.77	1168.31	1169.17	1172.33	1176.85	1119.88	1141.95	1140.26
176	352.813	306.436	306.376	1157.16	1154.15	1155.16	1158.45	1163.27	1098.84	1123.45	1121.84
177	365.357	308.244	308.094	1137.95	1134.33	1135.14	1139.11	1143.51	1069.65	1096.67	1095.16













	Average Mass Flow Rate, Part Turbo-Comp	Average Efficiency, Part Turbo-Comp	Average Power, Part Turbo-Comp	Average Mass Flow Rate, Part Turbo-Turbine	Average Efficiency, Part Turbo-Turbine	Average Power, Part Turbo-Turbine	Average Pressure (Outlet), Part SR-DK1	Average Pressure (Inlet), Part SR-DK2	Average Mass Flow Rate, Part Throttle-01	Orifice Diameter (End of Cycle), Part Turbo-WG-01
Index	kg/s	%	kW	kg/s	%	kW	bar	bar	No Unit	mm
96	0.0197859	40.2	0.279982	0.0211859	49.019	0.279719	1.04148	0.345932	19.7859	7.97885E-07
97	0.0250615	46.4326	0.550199	0.0268245	51.4603	0.550713	1.09491	0.420051	25.0615	7.97885E-07
98	0.0303548	52.8717	0.931593	0.0324855	52.9044	0.93243	1.17007	0.495317	30.3548	7.97885E-07
99	0.0356301	59.1393	1.3935	0.0381312	52.7242	1.39479	1.26096	0.569897	35.6302	0.1
100	0.0408866	61.3405	1.88759	0.0432049	52.6108	1.88603	1.33159	0.642927	40.8866	3
101	0.0461472	63.0591	2.23267	0.0464637	52.4727	2.23445	1.36005	0.714063	46.142	6.69
102	0.0513737	64.9662	2.39036	0.0479547	52.0647	2.39119	1.35174	0.783143	51.3737	10.18
103	0.0566537	66.3911	2.52047	0.049205	51.5914	2.52027	1.33697	0.852267	56.6537	12.69
104	0.0618701	67.4062	2.73732	0.0511369	51.2345	2.73689	1.33621	0.921504	61.8701	14.1834
105	0.0669305	68.2203	2.90924	0.0528931	50.9247	2.9075	1.32834	0.989387	66.9305	15.4001
106	0.0724146	69.1273	3.41953	0.0571756	51.155	3.42688	1.367	1.06277	72.4146	15.5261
107	0.0780634	69.6192	3.53855	0.0583753	50.603	3.53541	1.3439	1.13434	76.0634	16.8204
108	0.0842058	70.4527	4.2992	0.0640608	50.8911	4.29804	1.40324	1.21776	84.2058	16.5745
109	0.0904077	70.1525	4.43189	0.0653752	50.3791	4.42986	1.37031	1.29268	90.4077	17.6806
110	0.0975916	70.7777	5.59482	0.0729452	50.9855	5.59303	1.45661	1.39123	97.5916	17.1419
111	0.105363	70.6313	6.53215	0.0786002	51.0379	6.52996	1.49632	1.49106	105.363	17.2932
112	0.113342	71.2549	8.17559	0.0873663	52.0037	8.17321	1.61057	1.6049	113.342	16.713
113	0.120553	71.3189	9.80616	0.0956478	52.8087	9.80359	1.71402	1.70799	120.553	16.1649
114	0.0229086	44.0397	0.436827	0.0245589	50.7978	0.437236	1.07233	0.36658	22.9086	7.97885E-07
115	0.0289404	51.2807	0.826922	0.0309953	52.8945	0.826915	1.14896	0.444139	28.9404	0.000000798
116	0.0351308	58.6229	1.35249	0.0375997	53.5388	1.35322	1.25213	0.523123	35.1308	0.2
117	0.0412999	61.4027	1.9409	0.0437699	53.8019	1.94102	1.33771	0.600731	41.2999	2.59916
118	0.047219	63.4463	2.2929	0.0471075	53.9217	2.29264	1.36218	0.672004	47.219	7.2
119	0.0530169	65.2185	2.58645	0.0497763	53.9439	2.58674	1.37251	0.740413	53.0169	9.99
120	0.0588579	66.5986	2.9346	0.0527539	53.9848	2.93467	1.38814	0.809448	58.8879	11.75
121	0.0646979	67.7653	3.23816	0.0553353	53.9509	3.23757	1.39317	0.878129	64.6979	13.3
122	0.0704818	68.8724	3.50275	0.0576858	53.9137	3.50197	1.39171	0.946001	70.4818	14.6
123	0.0763513	69.7985	3.80781	0.0604104	53.8738	3.80726	1.39378	1.01494	76.3513	15.5334
124	0.0822786	70.1492	3.86411	0.0614721	53.6485	3.86487	1.35788	1.08043	82.2786	16.8801
125	0.0887569	70.6541	4.62065	0.0670153	53.7967	4.62011	1.40799	1.15843	88.7569	16.7771
126	0.0953723	69.762	4.66018	0.067906	53.4836	4.65883	1.35661	1.22726	95.3724	18.0133
127	0.102834	70.2909	5.75029	0.0750212	53.7533	5.74894	1.4262	1.3196	102.834	17.6806
128	0.110985	68.4372	6.3341	0.078986	53.6401	6.33249	1.40831	1.40142	110.985	18.2198
129	0.120448	69.645	8.20476	0.0889449	54.5848	8.20306	1.52784	1.52092	120.448	17.6386
130	0.130798	69.9501	10.4954	0.0999942	55.3674	10.4932	1.65728	1.64978	130.798	17.0638
131	0.0260572	48.0522	0.643255	0.027982	51.8212	0.643354	1.11557	0.381994	26.0572	0.1
132	0.0327744	56.0317	1.1652	0.0351347	53.3132	1.165	1.21832	0.460937	32.7744	0.180046
133	0.03957	61.1007	1.75269	0.0415219	53.7343	1.75215	1.31428	0.539578	37.957	3.78005
134	0.0463777	63.1577	2.23958	0.0461719	54.1319	2.23895	1.35964	0.61646	46.3777	7.3103
135	0.0530804	65.0975	2.71621	0.0504095	54.2908	2.71606	1.39635	0.691724	53.0804	9.55
136	0.0593615	66.9043	2.88428	0.0505545	54.127	2.88383	1.34498	0.755178	59.3614	13.3256
137	0.0660188	68.0522	3.24007	0.0552555	54.3286	3.24080	1.38541	0.828815	66.0188	13.9367
138	0.0723539	68.8486	3.08331	0.0547573	53.9565	3.08574	1.31774	0.891376	72.3539	16.4443
139	0.0791182	69.8133	3.67618	0.0596216	54.135	3.67767	1.35574	0.966537	79.1182	16.65
140	0.08578	70.1218	3.96377	0.0622588	54.0269	3.96601	1.34664	1.03659	85.78	17.5
141	0.0925843	69.4696	4.26277	0.0649196	53.918	4.25855	1.3304	1.10467	92.5843	18.2328
142	0.099767	67.8495	4.54469	0.067501	53.7622	4.54367	1.3027	1.16788	99.767	18.9499
143	0.108182	67.7068	5.54459	0.0743418	53.905	5.54426	1.34609	1.25657	108.182	18.7966
144	0.116783	67.4979	6.64912	0.0813787	54.1438	6.64881	1.38848	1.34594	116.783	18.6497
145	0.125783	67.5009	8.08148	0.089561	54.8087	8.08059	1.45001	1.44214	125.783	18.3651
146	0.13638	68.0407	10.4928	0.101552	55.7539	10.4918	1.57935	1.57086	136.38	17.6494
147	0.149119	67.8636	13.7853	0.115917	56.6538	13.7835	1.73652	1.72728	149.119	16.9041
148	0.0307622	50.8496	0.754961	0.0297864	51.9981	0.754576	1.12103	0.387506	30.7622	8.5
149	0.0379425	58.3436	1.22907	0.0360141	53.448	1.22987	1.20137	0.471971	37.9426	9.36403
150	0.0452129	63.7703	1.65807	0.0407695	53.9244	1.65864	1.25811	0.545482	45.213	11.3113
151	0.0527376	65.6841	2.24371	0.0464683	54.3428	2.24269	1.31714	0.622122	52.7376	12.2
152	0.0601453	67.0766	2.70365	0.0506876	54.5045	2.69931	1.34274	0.695434	60.1453	13.6633
153	0.0673585	68.3399	3.07461	0.0540848	54.567	3.07002	1.35085	0.766882	67.3585	15.0362
154	0.0748	69.5574	3.74186	0.0593963	54.7743	3.72941	1.39691	0.842787	74.8001	15.4486
155	0.0825414	70.3276	4.48792	0.0651576	55.0331	4.49788	1.44076	0.920588	82.5413	15.708
156	0.0896352	70.1589	4.24176	0.0644316	54.5286	4.24259	1.35308	0.982915	89.6353	17.7532
157	0.0969113	69.2287	4.57356	0.0673354	54.4483	4.57239	1.334	1.05182	96.9113	18.4627
158	0.104379	67.2389	4.95096	0.0705185	54.3622	4.94951	1.30975	1.12213	104.38	19.0337
159	0.11229	65.1474	5.47606	0.0746865	54.314	5.47426	1.29336	1.19793	112.29	19.3796
160	0.120872	64.5591	6.50662	0.0815668	54.5765	6.50505	1.32179	1.28418	120.872	19.2281
161	0.130882	65.1969	8.17335	0.0910635	55.3533	8.17222	1.39575	1.38637	130.882	18.7873
162	0.141315	65.8892	10.5938	0.103539	56.2859	10.5925	1.51496	1.50526	141.314	18.008
163	0.151445	66.2651	13.2755	0.118318	57.1356	13.2736	1.64298	1.63256	151.445	17.1988
164	0.0357472	59.601	1.48578	0.0384882	53.3891	1.4872	1.28263	0.423574	35.7471	0.016216
165	0.0435368	61.9747	2.14966	0.045147	54.3659	2.14698	1.36179	0.499932	43.5371	5.08675
166	0.0513427	64.669	2.57583	0.049047	54.8573	2.56938	1.38328	0.571304	51.3431	9.40594
167	0.0593058	66.7455	2.89561	0.0520372	55.0482	2.88842	1.37841	0.641412	59.306	12.5169
168	0.0673516	68.3405	3.16324	0.0546935	55.085	3.1569	1.36289	0.7115	67.3514	14.7905
169	0.0753858	69.3561	3.42391	0.0573396	55.0423	3.41912	1.34479	0.781409	75.3858	16.4266
170	0.0834536	69.7525	3.69857	0.0601032	54.972	3.69469	1.32573	0.852122	83.4537	17.6425
171	0.0916276	68.8844	4.0256	0.0632174	54.943	4.02162	1.30438	0.924434	91.6275	18.5345
172	0.100529	70.2326	5.38999	0.0725173	55.5454	5.38871	1.40373	1.012	100.529	17.8481
173	0.108759	67.377	5.52672	0.0744695	55.3175	5.52529	1.33632	1.08358	108.759	18.8715
174	0.11704	63.9867	5.85324	0.077436	55.1681	5.85192	1.28603	1.15734	117.04	19.522
175	0.125931	61.793	6.84242	0.0828892	55.2855	6.84187	1.27586	1.24132	125.931	19.8835
176	0.136035	62.8573	8.44895	0.0932522	56.144	8.44772	1.35388	1.34432	136.035	19.1156
177	0.147064	63.8052	11.0299	0.106884	56.9822	11.0285	1.47274	1.46236	147.064	18.2456







	Average Outlet Temperature, Part Turbo-Comp	Mass Averaged Temperature (Outlet), Part After-Cooler	Mass Averaged Temperature, Part SR-TV-23	Mass Averaged Temperature (Outlet), Part Exh_Manifold 1	Mass Averaged Temperature (Outlet), Part Exh_Manifold 2	Mass Averaged Temperature (Outlet), Part Exh_Manifold 3	Mass Averaged Temperature (Outlet), Part Exh_Manifold 4	Average Inlet Temperature, Part Turb80-1	Average Outlet Temperature, Part Turb80-1	Mass Averaged Temperature, Part Cat-IN	Mass Averaged Temperature, Part Cat-Out
Index	K	K	K	K	K	K	K	K	K	K	K
98	294.446	297.808	300.041	1140.12	1159.9	1155.72	1141.31	1116.37	1102.54	1089.29	1086.48
99	294.387	297.84	299.043	1151.35	1165.72	1160.53	1151.06	1127.79	1115.2	1101.95	1099.22
100	294.296	297.862	298.7	1158.35	1170.32	1165.21	1158.14	1139.07	1123.05	1111.15	1108.51
101	294.199	297.841	298.516	1165	1177.09	1172.27	1165.18	1149.72	1129.76	1119.26	1116.68
102	294.327	297.837	298.414	1172.68	1185.68	1181.36	1173.07	1160.26	1136.1	1126.77	1124.27
103	294.318	297.827	298.352	1181.15	1193.59	1190.05	1181.84	1169.98	1141.34	1133.06	1130.65
104	294.517	297.836	298.325	1191.03	1201.59	1198.76	1191.85	1179.65	1146.27	1138.91	1136.61
105	295.021	297.872	298.334	1200.12	1211.36	1209.23	1200.89	1189.44	1150.97	1144.49	1142.29
106	295.699	297.923	298.375	1207.24	1222.75	1221.56	1207.76	1198.81	1155.15	1149.49	1147.4
107	297.227	298.033	298.436	1203.23	1221.24	1222.81	1203.3	1197.18	1148.82	1144.22	1142.24
108	307.487	298.974	299.226	1198.99	1215.85	1217.76	1198.81	1192.86	1139.18	1135.78	1133.92
109	316.121	299.829	299.964	1196.17	1212.94	1215.52	1195.96	1190.33	1130.74	1128.52	1126.77
110	322.499	300.517	300.569	1194.49	1211.32	1214.4	1194.3	1188.81	1123.02	1121.95	1120.29
111	330.502	301.409	301.365	1195.75	1212.74	1216.33	1195.48	1189.99	1117.37	1117.43	1115.85
112	338.731	302.38	302.24	1201.32	1218.79	1222.83	1201	1195.34	1114.85	1116.05	1114.55
113	347.891	303.512	303.269	1209.06	1227.01	1231.53	1208.58	1202.72	1113.46	1115.87	1114.44
114	356.697	304.658	304.321	1213.46	1231.6	1236.58	1212.62	1206.85	1109.16	1112.81	1111.45
115	364.781	305.745	305.324	1210.39	1228.55	1233.72	1209.77	1204.06	1099.52	1104.33	1103.03
116	294.428	298.663	300.102	1146.02	1185.07	1181	1147.58	1132.32	1120.75	1106.78	1103.89
117	294.323	298.705	299.56	1160.26	1185.81	1181.51	1161.82	1144.47	1130.27	1117.4	1114.69
118	294.21	298.686	299.274	1171.79	1185.57	1181.08	1173.34	1154.97	1136.78	1125.51	1122.88
119	294.441	298.681	299.148	1179.9	1188.64	1184.6	1181.81	1164.26	1141.58	1131.83	1129.28
120	294.559	298.676	299.096	1186.86	1195.59	1192.17	1188.61	1173.48	1146.05	1137.53	1135.09
121	294.48	298.661	299.063	1194.23	1205.18	1202.42	1195.52	1183.18	1150.74	1143.23	1140.91
122	295.096	298.704	299.084	1201.82	1215.7	1213.59	1202.88	1192.88	1155.07	1148.49	1146.29
123	295.54	298.739	299.111	1206.74	1225.96	1224.56	1207.34	1200.84	1157.45	1151.8	1149.73
124	296.659	298.829	299.19	1207.4	1227.23	1228.64	1207.4	1202.79	1153.89	1149.3	1147.33
125	299.936	299.09	299.414	1204.73	1221.61	1223.39	1203.68	1199.04	1144.54	1141.14	1139.3
126	311.459	300.22	300.404	1199.17	1214.87	1217.1	1198.15	1193.57	1132.9	1130.78	1129.05
127	318.828	301.019	301.109	1195.77	1211.14	1213.75	1194.86	1190.45	1123.18	1122.29	1120.66
128	326.405	301.885	301.884	1194.06	1209.41	1212.42	1193.19	1188.99	1114.51	1114.85	1113.3
129	334.543	302.862	302.772	1193.74	1208.93	1212.28	1192.77	1188.71	1106.34	1107.95	1106.47
130	343.598	304.002	303.818	1195.55	1211.37	1215.13	1194.59	1190.88	1099.58	1102.53	1101.11
131	353.227	305.28	304.998	1200.25	1216.7	1220.85	1199.25	1195.83	1094.45	1098.84	1097.48
132	363.534	306.709	306.315	1205.95	1223.33	1227.77	1204.75	1201.93	1089.94	1095.83	1094.51
133	373.194	308.103	307.589	1208.38	1223.99	1228.6	1207.79	1203.89	1082.39	1089.72	1088.45
134	294.374	299.527	300.748	1164.47	1175.27	1172.59	1165.76	1138.17	1124.96	1111.76	1108.96
135	294.269	299.544	300.352	1181.54	1193.2	1190.27	1181.95	1159.14	1142.51	1130.22	1127.56
136	294.161	299.508	300.119	1196.64	1209.22	1206.2	1196.7	1178.63	1156.99	1146.08	1143.47
137	294.769	299.527	300.018	1204.9	1218.57	1214.82	1206.28	1190.84	1163.75	1154.3	1151.79
138	294.136	299.464	299.893	1209.48	1223.61	1220.79	1211.29	1198.2	1165.5	1157.39	1155.01
139	294.341	299.468	299.863	1216.06	1230.11	1228.75	1216.52	1206.02	1167.41	1160.47	1158.22
140	294.89	299.508	299.875	1216.94	1231.1	1230.55	1217.03	1208.19	1163.55	1157.86	1155.74
141	295.441	299.554	299.913	1214.34	1228.64	1228.71	1214.29	1206.76	1156	1151.61	1149.61
142	297.478	299.714	300.086	1214	1225.1	1225.52	1212.26	1205.18	1147.97	1144.87	1142.97
143	304.466	300.384	300.672	1214	1223.01	1223.67	1212.02	1204.63	1140.45	1138.64	1136.82
144	315.906	301.598	301.748	1210.18	1219.8	1220.97	1208.62	1201.95	1130.48	1130.01	1128.26
145	323.406	302.465	302.499	1206.58	1216.44	1218.04	1205.37	1199.19	1120.08	1120.95	1119.28
146	331.824	303.49	303.426	1206.7	1216.71	1218.74	1205.53	1199.84	1113.96	1113.96	1112.35
147	340.551	304.603	304.441	1198.55	1208.3	1210.68	1197.39	1192.42	1095.72	1099.54	1098
148	349.663	305.816	305.56	1187.31	1196.42	1199.11	1186.21	1181.88	1076.87	1082.22	1080.76
149	360.349	307.292	306.93	1182.79	1191.7	1194.73	1181.71	1177.99	1063.13	1070.08	1068.67
150	372.949	309.108	308.625	1188.05	1196.84	1200.16	1186.84	1183.5	1056.76	1065.42	1064.06
151	389.291	311.515	310.884	1200.82	1209.58	1213.24	1199.43	1196.36	1056.33	1066.83	1065.49
152	294.371	299.277	300.572	1178.43	1174.62	1172.4	1179.73	1147.36	1131.68	1119.52	1116.88
153	294.14	299.232	299.931	1196.09	1193.36	1190.62	1196.82	1168.63	1148.57	1137.49	1134.93
154	294.043	299.2	299.623	1212.49	1209.77	1206.94	1213.26	1188.19	1162.36	1152.62	1150.13
155	293.929	299.173	299.48	1223.13	1220.98	1219.02	1224.14	1201.9	1169.69	1161.35	1158.96
156	293.951	299.161	299.443	1226.83	1226.18	1224.34	1227.66	1208.29	1169.49	1162.6	1160.32
157	294.063	299.164	299.473	1224.74	1225.46	1223.98	1225.73	1208.65	1163.25	1157.84	1155.71
158	294.712	299.223	299.569	1220.02	1221.7	1220.97	1220.96	1205.93	1153.73	1149.81	1147.77
159	295.634	299.311	299.722	1214.5	1217.39	1217.03	1215.15	1202.14	1142.83	1140.38	1138.45
160	301.102	299.811	300.309	1211.43	1214.48	1214.51	1211.32	1199.85	1133.08	1132.05	1130.18
161	313.637	301.142	301.459	1211.04	1214.6	1214.72	1211.09	1200.32	1125.46	1125.76	1123.99
162	322.004	302.113	302.317	1209.26	1213.03	1213.33	1209.33	1199.25	1116.35	1117.99	1116.28
163	330.08	303.107	303.217	1200.96	1204.71	1205.2	1201.13	1191.89	1101	1104.09	1102.43
164	338.535	304.198	304.216	1187.73	1191.14	1191.9	1187.87	1179.65	1080.68	1085.3	1083.73
165	347.903	305.463	305.386	1175.74	1178.83	1179.77	1175.96	1168.62	1060.53	1066.75	1065.26
166	358.422	306.931	306.762	1159.34	1161.86	1163.03	1159.7	1153.21	1036.35	1044.21	1042.82
167	369.368	308.494	308.251	1137.38	1138.98	1140.24	1137.44	1132.05	1008.1	1017.46	1016.15
168	381.42	310.282	309.938	1125.1	1126.06	1127.49	1125.16	1120.36	988.351	999.156	997.913
169	294.239	299.247	300.706	1186.06	1181.77	1176.89	1185.48	1155.56	1137.32	1126.1	1123.56
170	294.204	299.218	300.115	1200.45	1195.25	1190.4	1199.76	1172.81	1149.44	1139.53	1137.07
171	294.232	299.199	299.767	1213.19	1207.67	1202.7	1212.86	1188.38	1158.53	1150.09	1147.71
172	293.736	299.146	299.569	1221.51	1215.74	1211.08	1221.46	1198.95	1161.94	1155.02	1152.73
173	293.961	299.156	299.535	1223.86	1218.97	1214.91	1224.23	1203.72	1159.16	1153.81	1151.64
174	294.018	299.161	299.545	1221.36	1217.6	1214.25	1222.19	1203.54	1151.27	1147.53	1145.47
175	294.292	299.192	299.634	1215.81	1212.72	1210.04	1216.81	1199.82	1139.74	1137.59	1135.64
176	295.324	299.3	299.856	1208.28	1205.8	1204	1209.4	1193.98	1125.96	1125.39	1123.53
177	309.4	300.8	301.273	1201.74	1199.04	1197.75	1202.68	1188.42	1111.48	1112.54	1110.77
178	319.976	302.019	302.375	1195.7	1191.72	1191.72	1196.62	1183.2	1097.44	1100.03	1098.34
179	328.479	303.074	303.34	1190.13	1187.2	1186.27	1191.32	1178.49	1084.07	1088.08	1086.47
180	337.205	304.214	304.394	1182.93	1179.48	1178.73	1183.93	1171.89	1068.62	1074.09	1072.54
181	346.848	305.528	305.624	1168.78	1164.85	1164.35	1169.72	1158.67	1046.56	1053.61	105















## Appendix D: Comparison of SuperTurbo test data to modeling data

Parameter	Engine Speed	BMEP	Torque	Power	IMEP(720)	IMEP(360)	PMEP	FMEP
UNIT	RPM	bar	N-m	kW	bar	bar	bar	bar
Test Data WG closed	2707	13.40	211.5	60.0	14.1	14.7	-0.54	0.73
TestData WG open	2707	13.30	210.0	59.5	14.3	14.7	-0.38	0.99
SuperTurbo Model	2707	13.40	211.6	60.0	14.5	14.8	-0.32	1.36
Stock Model	2707	13.40	211.5	60.0	14.8	15.0	-0.21	1.36

Parameter	Atmospheric Pressure	Atmospheric Temperature	Compressor Outlet Pressure	Turbine Inlet Temperature	Turbine Outlet Temperature	Turbine Inlet Pressure	Turbine Outlet Pressure	Catalyst Inlet Pressure
UNIT	bar	K	bar	K	K	bar	bar	bar
Test Data WG closed	1.00	302	1.34	1117	980	1.72	1.10	1.06
TestData WG open	1.00	302	1.33	1114	980	1.54	1.10	1.06
SuperTurbo Model	1.00	302	1.39	1152	1102	1.69	1.17	1.17
Stock Model	1.00	302	1.39	1155	1114	1.61	1.19	1.17

Parameter	Catalyst Outlet Pressure	Lambda	BSFC	Fuel Injection Rate	Inlet Air Mass Flow Rate	Wastegate Flow	Wastegate Signal	Turbine Speed
UNIT	bar	No Unit	g/kW-h	g/s	g/s	g/s	%	RPM
Test Data WG closed	1.01	0.97	248	1.03	58.6	0	100	90486
TestData WG open	1.01	0.97	250	1.03	58.6	NA	29	89562
SuperTurbo Model	1.09	0.96	245	1.02	56.1	0	NA	90486
Stock Model	1.10	0.96	248	1.03	56.8	11.6	NA	90276

Parameter	50% Burned Crank Angle	Ignition Timing	CVT Torque added to Crankshaft	Calculated Gear Ratio
UNIT	deg	deg	N-m	No Unit
Test Data WG closed	14.55	-13.19	4.31	33.427
TestData WG open	14.57	-13.21	0.46	33.085
SuperTurbo Model	12.90	-13.19	4.51	33.427
Stock Model	12.90	-13.19	NA	33.349