

Performance of a Wind Turbine Blade in Sandstorms Using a CFD-BEM Based Neural Network

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Abstract

In arid regions, such as the North African desert, sandstorms impose considerable restrictions on horizontal axis wind turbines (HAWT) which have not been thoroughly investigated. This paper examines the effects of debris flow on the power generation of HAWT. Computational Fluid Dynamics (CFD) models were established and validated to provide novel insights on the effects of debris on the aerodynamic characteristics of NACA 63415. To account for the change in chord length and Reynolds number along the span of the blade and the 3D flow patterns, the power curves for a wind turbine were obtained using the Blade Element Momentum (BEM) method. We present a novel coupled application of neural network, CFD and BEM to investigate the erosion rates of the blade due to different sandstorm conditions. The proposed model can be scaled and developed to assist in monitoring and prediction of HAWT blade conditions. This work shows that HAWT performance can be significantly diminished due to the aerodynamic losses under sandstorm conditions. The power generated under debris flow conditions can decrease from 10 to 30 % compared to clean conditions.

Keywords: wind energy; wind turbine blade aerodynamics; debris flow; surface roughness; CFD-BEM model; neural network

Nomenclature

a	Axial Induction Factor (-)
\hat{a}	Tangential Induction Factor (-)
AOA	Angle of Attack ($^{\circ}$)
b	Number of Blades (-)
c	Airfoil Chord Length (m)
C_d	Drag Coefficient (-)
C_l	Lift Coefficient (-)
C_m	Torque Coefficient (-)
C_n	Normal Load Coefficient (-)
C_p	Power Coefficient (-)
C_t	Tangential Load Coefficient (-)
D_p	Particle Drag Force Coefficient (-)
d_p	Sand Particle Diameter (m)
dF_D	Element Drag (N)
dF_L	Element Lift (N)
dF_N	Element Normal (N)
dF_T	Element Tangential (N)
dM	Torque (Nm)
dT	Thrust Force (N)
m_x	Additional Particle Acceleration (m/s^2)
P	Rotor Shaft Power (watt)
r	Local Radius (m)
Re	Reynolds Number (-)
Re_p	Relative Reynolds Number for the Sand Particle (-)
Re_{λ}	Microscale Reynolds Number (-)
$R_{erosion}$	Particle erosion (g/m^2)
V_0	Real Value of Velocity (m/s)
V_p	Particle Velocity (m/s)
V_{rel}	Relative Velocity (m/s)
Y^+	Dimensionless Wall Distance (-)
α	Characteristic Ratio Between the Sand and Air (-)
μ	Fluid Dynamic Viscosity ($kg/m\ s$)
ρ	Fluid Density (kg/m^3)
ρ_p	Sand Particle Density (kg/m^3)
Φ	Relative Wind Angle ($^{\circ}$)
ω	Rotational Speed (rad/s)

1.0 Introduction

In aeronautics, the determination of airfoil performance is vitally important. Computational Fluid Dynamics (CFD) and Blade Element Momentum (BEM) approaches are the leading methods used to simulate wind turbine blade performance. The BEM method analyses the flow field and blade aerodynamics, to calculate the rotor shaft torque and maximize the power generated [1-5]. BEM is fast with low computational cost, and implementation is relatively simple. However, CFD is more accurate and provides more detailed results, but incurs higher computational cost [5]. The CFD-BEM mixed approach was used in designing HAWT blades and predicting the wind turbine performance [5]. For example, Esfahanian et al. [6] determined the aerodynamic coefficients of span wise 2D sections of NREL Phase II wind turbine blades using CFD and then use BEM to predict the turbine performance. The CFD-BEM mixed approach had a much lower computational costs than the CFD-only approach yet a high degree of accuracy was obtained, which was verified using experimental results. Yang et al. [7] used a 2D CFD simulation to derive the lift and drag coefficients. The extracted airfoil data were input directly into a BEM code, which was firstly compared with experimental data for the axial and tangential forces on the blade. Then, they used these derived airfoil data sets to determine the axial and tangential forces for different blade pitch and wind velocity. Good agreement was obtained compared to experiments. On the other hand, the CFD-BEM model has also been used for other applications such as tidal stream turbines [8] and marine current turbines [9].

Research concerning the performance and deficit in energy yield of horizontal axis wind turbines (HAWT) for locations subject to dust and sand abrasion is still incomplete, while arid regions are a key focus for further development of large scale wind farms [10-12]. Akour et al. [13] used BEM theory for the blade design of airfoils BW3, A18 and SG6043. To account for changes in the blade chord length along the span and 3D flow patterns, the power coefficients of each blade was obtained using the software package QBlade. The simulation results were validated using a prototype tested in open air environment. Also, Darbandi et al. [14] obtained the blade aerodynamics coefficients using

1 CFD simulation after validating against experimental data for a 1 megawatt wind turbine airfoil. The
2 CFD results proved that the blade roughness could effectively reduce the aerodynamic coefficients of
3 a clean airfoil. BEM theory was used to predict the performance of the 1-megawatt wind turbine blade.
4 Results showed that due to the rough surface the 1-megawatt wind turbine could be faced with 25%
5 reductions in its annual energy production. In addition, Pechlivanoglou et al. [15] studied the
6 aerodynamic effects of various types of roughness-related shape deviations on wind turbines using
7 XFOil investigations. The XFOil simulation results were validated using wind tunnel measurements.
8 Measurements of power produced by wind turbines operated in sandy conditions were also used to
9 determine the actual effects of rough surfaces. Power predictions made using the state BEM method
10 were correlated with the actual power measured.

11 Artificial neural network (ANN) algorithm was used for wing sections and airfoils performance
12 optimization in various research studies. Important contributions are made by ANN in the airfoil
13 designs and the tip speed ratio (TSR) selection. For instance, Chen and Agarwal[16] proposed a genetic
14 algorithm with an ANN to optimize the wind turbines' flatback airfoils. The technique was proved to
15 find the optimal flatback airfoils. A neuro-fuzzy inference system was introduced by Ata and Kocyigit
16 [17] in application of wind turbine in order to estimate its TSR and power factor. Proposed neuro-
17 fuzzy inference system was shown by the model that it improves the conventional methods
18 performance. Yurdusev et al. [18] investigated the optimum TSR of the wind turbine airfoil designs
19 hugely used in practice as well. A demonstration of a multi-layer feed-forward neural network-based
20 model was done. The ANN proposed model's results proved that it is fast and accurate. It showed that
21 the algorithm can be modified into other airfoil designs easily due to the neural networks'
22 generalization and adaptable capabilities. In Mortazavi et al. [19] , the airfoil design for the blade
23 sections of HAWT were done. They used computational fluids dynamics to train their ANN algorithm
24 in order to obtain a Pareto optimal set of solutions for the airfoil section's geometrical characteristics.
25 Some studies in literature applied ANN for fault classification were carried out in the past decades,

1 where the monitored component could be discovered by the method whether it is faulty or not.
2 Saravanan and Ramachandran[20] for instance, developed an ANN with a high potential in monitoring
3 the fault conditions of the gear box. Moreover, a multi-layer back propagation neural network-based
4 model was used by Momoh and Button[21] for detecting any Direct Current arcing faults in a
5 spacecraft used by NASA in its experimental set up. The operator can only know if the component
6 was failed or not using that way. Fault development or estimation cannot be tracked by the Operator.
7 On the other hand, the methodology to automatically predict early faults of wind turbine main bearings
8 was shown by Zhang [22] . This is done by analyzing SCADA data based on ANN.

9 The main aim of the current study is to analyze HAWT performance and the energy yield
10 deficit due to debris flow using CFD-BEM modelling. Also, a new technique is proposed using ANN
11 to predict the amount of erosion occurred at different debris flow conditions. The results will provide
12 wind turbine designers with a method to estimate the performance of wind turbines installed in dusty
13 environments and its change over time.

14 **2.0 CFD-BEM Model**

15 The 3D wind turbine performance is predicted using QBlade software, because its results have
16 been verified by several researchers by comparison with wind tunnel test data and results from full
17 scale wind turbines [23]. QBlade software uses a BEM code that is based on an algorithm developed
18 by Hansen [24].

19 In the presented study, QBlade software was used to compute the aerodynamics of a wind
20 turbine made of three blades of 20.5 meters in length. Figure 1 presents the layout of the wind turbine
21 blade and the airfoil used which is NACA 63415. Individual data sets for clean and rough conditions
22 due to debris flow were prepared according to the results obtained by CFD. These results represent the
23 lift and drag coefficients of the chosen airfoil over an angle of attack range from 0° to 10°.

1 The basic assumptions used follow from the assumptions of the BEM theory, which is
2 summarized as:

- 3 • The blade is discretized into segments.
- 4 • Any aerodynamic interaction between segments is neglected.
- 5 • Lift and drag forces on the blades are determined using the airfoil characteristics.
- 6 • The wind turbine performance is determined by the BEM method using the Qblade
7 software Package.
- 8 • While the aerodynamic interaction is neglected in determining the lift and drag coefficients,
9 Qblade makes a Glauert correction for the aerodynamic effects of the neighboring elements.

10 In the current study, a control approach, maximum power point tracking (MPPT), is used to
11 determine the optimal tip speed ratio (TSR) to use in the operation of the stall-regulated system [5].

12 13 **2.1 CFD Model Description and Simulation Details:**

14 Reynolds-averaged Navier-Stokes (RANS) equations were used to simulate two-dimensional,
15 viscous, incompressible flow. The continuity equation and momentum equation based on RANS
16 equations are:

$$17 \quad \frac{\partial u_i}{\partial x_i} = 0 \quad (1)$$

$$18 \quad \frac{\partial}{\partial t}(\rho u_i) + \frac{\partial}{\partial x_j}(\rho u_i u_j) = \frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) - \overline{\rho u_i' u_j'} - \alpha \rho_p \frac{\partial V_{pi}}{\partial t} \right] \quad (2)$$

19 where α is the sand to air characteristic ratio, ρ_p is the particle density and V_p is the particle velocity.

20 CFD simulations for clean air flow conditions were run for comparison with wind tunnel
21 experimental data taken from [25] to verify the numerical model for the free-stream flow over the
22 NACA 63415 airfoil. The experiments were conducted at a Reynolds number of 1.6×10^6 .

23 A comparison between the numerical results at Reynolds numbers of 1.6×10^6 , 460×10^3 and
24 300×10^3 and the experimental data of the lift (C_l) and drag (C_d) coefficients variation with the angle
25 of attacks (AOA) have been investigated using different turbulence models. This is shown in figure 2.

1 As shown in figures in figure 2 (a) and (b), the numerical results of SST k- ω model had good
2 agreement with the experimental data at Reynolds number 1.6×10^6 , except for small deviations in the
3 lift coefficient. On the other hand, in figure 2 (c), (d), (e) and (f), the transition SST model gave the
4 closest match with the expected data at Reynolds numbers of 460×10^3 and 300×10^3 . Since separated
5 flow is directly connected to flow in the boundary layer, the transition SST turbulence model should
6 best capture this critical phenomena for wind turbine applications [26]. Similar results have been found
7 for other airfoils typically used with wind turbines, such as S822 [28].

8 As shown in figure 2 (e) and (f), using the transition SST turbulence model, the drag coefficient
9 values along the angle of attack variation at Reynolds number 300×10^3 are much higher compared to
10 the reference experimental data. However, the lift coefficient has lower values along the variation of
11 the angle of attack. This is due to the transitional separation bubble phenomenon. According to the
12 study presented by [27] for NACA 63415 airfoil, a transitional separation bubble occurs when the flow
13 over the airfoil experiences decreasing pressure causing it to separate from the surface. After the flow
14 separates, a detached shear layer form. A transition to turbulent flow occurs within the unstable shear
15 layer. In the turbulent flow, momentum transfer is enhanced, which leads to reattachment. The size of
16 the transitional separation bubble increases with decreasing Reynolds number. The boundary layer
17 forms over the top of the separation bubble and therefore the airfoil drag increases substantially. The
18 transitional separation bubble and thickening of the boundary layer also affects the airfoil lift.

19 As presented in the authors' previous study in reference [29], the grid contained 120,878 nodes,
20 and the height of the grid next to the airfoil surface was 7×10^{-6} m. The dimensionless wall distance
21 Y plus (Y^+) values were less than 1 over the entire airfoil surface.

22 After validating the 2D CFD simulation model at the three Reynolds number values with the
23 experimental results from [25] in clean air, the Discrete Phase Model was utilized to predict the effect
24 of sand particles concentration and angle of attack on the erosion rate of the blade. The sand particle
25 diameter (d_p) for this study was selected as $250 \mu\text{m}$ with a sand density of 2500 kg/m^3 based on studies

1 in the Arabian Peninsula and Southern Africa [30, 31]. Three mass flow rate values were studied
 2 during the CFD simulation. These values are 100 kg/s, 200 kg/s and 400 kg/s. Each mass flow value
 3 is equivalent to a certain value of the characteristic ratio (α) between sand and air flow.

4 The Discrete Phase Model represents the sand particles in the continuous phase using round
 5 particles. The trajectories, heat transfer and mass transfer of these discrete phase entities are computed
 6 and simulated. Full coupling between the phases is included. The trajectory of the sand particle is
 7 predicted by integrating the force balance on the particle. This force balance is presented in a
 8 Lagrangian reference frame. The balance of the forces acting on the particle and the particle inertia,
 9 can be presented as:

$$11 \frac{dV_p}{dt} = [D_p(V_{rel} - V_p)] + \left[\frac{g_x(\rho_p - \rho)}{\rho_p} \right] + m_x \quad (3)$$

$$12 D_p = \frac{18 \mu C_d Re_p}{\rho_p d_p^2 24} \quad (4)$$

13
 14 where, $D_p (V_{rel} - V_p)$ is the drag force per unit particle mass, m_x is the virtual mass flow which is
 15 neglected since the density ratio between air to sand is very small, V_{rel} is the fluid velocity relative to
 16 the airfoil, μ is the fluid dynamic viscosity coefficient, ρ is the air density, ρ_p is the density of the sand
 17 particle, d_p is the sand particle diameter, Re_p is the relative Reynolds number for the sand particle [32].

18
 19 The characteristic ratio (α) between sand and air flow is

$$20 \alpha = \frac{\text{Volumetric Flow Rate of Sand}}{\text{Volumetric Flow Rate of Air}} \quad (5)$$

21 The erosion rate from particle impact is calculated for wall surfaces. The erosion rate is
 22 calculated as:

$$23 R_{Erosion} = \sum_{p=1}^{N_{Particles}} \frac{\dot{m}_p C(d_p) f(\theta) v^{b(v)}}{A_{face}} \quad (6)$$

24 Where \dot{m}_p is the mass flow rate of the injected discrete particles, $C(d_p)$ is a function of sand
 25 particle diameter, θ is the impact angle of the particle path with the wall face, $f(\theta)$ is a factor

1 which is a function of impact angle, v is the relative particle velocity, $b(v)$ is a function of relative
2 particle velocity, and A_{face} is the area of the cell face at the wall. Default values are $C= 1.8 \times 10^{-9}$,
3 $f=1$, and $b=0$ [32-34]. C , f and b values for sand eroding are given by Edwards et al.[35].

4 According to Lain and Sommerfeld [36], a multiphase flow simulation could be investigated
5 using one-way coupling in sand/air flow cases with a characteristic ratio (α) up to 6.3×10^{-4} . Lain and
6 Sommerfeld [36] found a good agreement between the turbulence modelling strategies used and the
7 experimental measurements by Tsuji et al. [37] in the multi-phase flow. Accordingly, the values of the
8 characteristic ratio (α) for each Reynolds number were chosen up to 6.3×10^{-4} . Tables 1, 2 and 3
9 represent the equivalent values of the characteristic ratio (α) for the three values of the mass flow rate
10 at each Reynolds number used in the CFD simulation.

11 Bose et al. [38], Monchaux and Dejoan [39] and Malloupas et al. [40] showed that two-way
12 coupling makes some difference to the results even for α in the range of 10^{-5} to 10^{-4} if $Re_\lambda < 50$, where
13 Re_λ is the Reynolds number based on the Taylor microscale length. However, Mora et al. [41] found
14 that two-way coupling has some effect on dissipation and settling time for the fluid droplets in their
15 study with Re_λ up to 400 or 500. For the flow conditions in the current study, Re_λ is in the range 3000
16 - 18000. Therefore, for Re_λ in the range used in the current study, one-way coupling should be
17 adequate. Further discussion on simulation of small droplets can be found in [42-43].

18 Mass flow rate 400 kg/s could not be simulated at Reynolds number of 300×10^3 as the value
19 of the characteristic ratio (α) in that case is higher than 6.3×10^{-4} , which is beyond the range verified
20 according to [36]. A high characteristic ratio would need a 4-way coupling model to include the particle
21 effects on turbulence and particle-particle interactions.

22 **2.2 Blade Element Momentum Model [24]**

23 BEM theory balances the axial force and moment generated on the rotor blades with changes
24 in, respectively, the linear and angular momentum of the mass of air flowing through the rotor disc.
25 This equilibration considers the flow segmented through annular elements of width dr as shown in
26

1 figure 3. In the represented BEM model, it was assumed there was no radial dependency and the force
2 from the blades on the flow within each annular element is uniform [24] . The data were adjusted to
3 360° polars based on the algorithms of Montgomerie and corrected for the tip losses (Prandtl's Loss
4 Factor correction). [15,44]. The elements near the root of the blade have angles of attack of greater
5 than 10 deg., so there is some loss of accuracy in extrapolating from the CFD data set. However, this
6 region accounts for only 15% of the power generated and has minimal erosion, so the overall accuracy
7 is only slightly affected.

8 The loads normal to the blade radius acting on the blade are shown in Figure 3. Also shown in
9 Fig. 3 are the angle of wind relative to the airfoil (Φ), the angle of attack (AOA), the axial and tangential
10 induction factors (a) and (\hat{a}) that significantly affect the real value of the velocity (V_0), and the element
11 normal (thrust) (dF_N) and element tangential (dF_T) forces which are generated by element lift (dF_l) and
12 drag (dF_d) forces.

13 2.2.1 Forces acting on each blade element:

$$15 \quad dF_N = \frac{1}{2} \rho V_{rel}^2 c C_n dr \quad (7)$$

$$16 \quad dF_T = \frac{1}{2} \rho V_{rel}^2 c C_t dr \quad (8)$$

17 Where, ρ is the air density, V_{rel} is the relative velocity, c is the chord length, C_l is the lift coefficient
18 and C_d is the drag coefficient,

$$19 \quad C_n = C_l \cos \Phi + C_d \sin \Phi \quad (9)$$

20 and

$$21 \quad C_t = C_l \sin \Phi - C_d \cos \Phi \quad (10)$$

22 From figure 3 it is readily seen from the geometry that:

$$23 \quad V_{rel} \sin \Phi = V_0(1 - a) \quad (11)$$

24 and

$$25 \quad V_{rel} \cos \Phi = \omega r(1 + \hat{a}) \quad (12)$$

1 Where, ω is the rotational speed and r is the local radius.

2 The Thrust force (dT) and the torque (dM) on the control volume of thickness dr are:

3
$$dT = b dF_N dr \quad (13)$$

4
$$dM = b dF_T r dr \quad (14)$$

5 Where, b is the number of wind turbine blades.

6 Using equation (11) for F_N and equation (15) for V_{rel} , equation (17) becomes:

7
8
$$dT = \frac{1}{2} \rho b \frac{V_0^2(1-a)^2}{\sin^2 \theta} c C_n dr \quad (15)$$

9 Similarly, if equation (12) is used for F_T and equations (15) and (16) are used for V_{rel} , equation (18)

10 becomes:

11
$$dM = \frac{1}{2} \rho b \frac{V_0(1-a)\omega r(1+a)}{\sin \theta \cos \theta} c C_t r dr \quad (16)$$

12
13 **2.2.2 Rate of change of momentum:**

14 Conservation of linear momentum is

15
$$dT = 4V_0^2 \rho \pi r a(1-a) dr \quad (17)$$

16 However, conservation of angular momentum is

17
$$dM = 4V_0 \rho \pi \omega r^3 \dot{a}(1-a) dr \quad (18)$$

18
19 Applying conservation of linear and angular momentum on a blade element of width dr , the
20 thrust force and moment supplied by a blade sector can be calculated from Equations (17) and (18).
21 The solution that simultaneously meets Equations (17), (18), (19) and (20) is found by iterating
22 respectively the axial and tangential induction factors (a) and (\dot{a}). The total moment applied at the
23 rotor shaft is found by summing the partial moments, dM , of each element of width dr .

24
$$P = \int \omega dM \quad (19)$$

25 By summing over the elements, the power at the rotor shaft (P) can determined using equation (19).

3.0 Erosion Prediction of NACA 63415 Using Neural Network

Using the back-propagation ANN, the total amount of erosion occurred per unit length for the HAWT blade made of NACA 63415 was estimated. Debris flow rate, Reynolds number and the angle of attack are the operating conditions (input data). In MATLAB software version R2015a, the proposed ANN was applied. Input, output and one or more hidden layers are there [45,46]. It was stated in Hertz et al. [47] and Goh's [48] work that the three layers ANN give credible results in most of the study cases. This statement was assured by the literature where ANNs were applied in process control. The three layered ANN was used in almost all of these cases[47,49-52]. Therefore, this proposed ANN consists of three-layer network, where it has in the hidden layer a sigmoid transfer function and in the output layer a linear transfer function.

The sigmoid transfer function $g(h)$ is

$$g(h) = \frac{1}{1+e^{-\beta h}} \quad (20)$$

Where, β is the rate constant. Three neurons form the input layer, only a single neuron forms the output layer and eight neurons are found in the hidden layer. Trial and error reveal the optimum number of hidden neurons. The implemented ANN in MATLAB software is shown in Figure 4 (a).

In a matrix, a set of input data were organized as columns. Into a second matrix, another set of target data (the correct output for each of the input data) were arranged. From the CFD study mentioned and discussed in the earlier section, these data were obtained. In table 4, the patterns used in the ANN training progress are shown. From the ANN training progress, eight patters were excluded. However, later on these patterns were used in the developed ANN to test its reliability. The excluded patterns are presented in Table 5. The patterns used in the ANN training progress were normalized because the sigmoid function's computed output can only be between 0 and 1.

Three sets were used where the ANN training progress were divided upon them randomly. The ratios were 85 % used for ANN training, 10 % used for ANN validation and the last 5 % used as a

1 completely independent test of network generalization. The ANN outputs with respect to targets (the
2 correct CFD output for each of the input data) are shown in figure 4 (b) for training, validation, and
3 test sets. The data should fall along 45-degree line for a perfect fit, which means that the targets are
4 equal to the ANN outputs.

5 **4.0 Results and Discussion**

6 **4.1 Effects of Debris Flow on Lift Coefficient (C_l) and Drag Coefficient (C_d)**

7 After the 2D CFD model was validated using experimental results for clean air flow, the
8 Discrete Phase Model was used to study the airfoil performance in different debris flow conditions.
9 The discrete phase particles were set to come out from the inlet with Reynolds number values of $1.6 \times$
10 10^6 , 460×10^3 and 300×10^3 .

11 As shown in figure 5 (a), (b) and (c), the lift coefficient value increases with an increase in
12 angle of attack due to the rise of the pressure difference between the high- and low-pressure sides for
13 both clean and rough conditions. As the amount of debris flow rate increases by increasing the debris
14 mass flow rate, the pressure coefficient difference between low- and high-pressure sides decreases as
15 the momentum transfer between the Lagrangian and Eulerian increased.

16 On the other hand, as shown in figure 5 (d), (e) and (f), the drag coefficient increases with angle
17 of attack, for both clean and rough conditions. As the angle of attack increases the drag coefficient
18 increases more in rough conditions due to debris than in clean conditions. The inclusion of sand
19 particles in the flow leads to a larger skin frictional drag and thus to larger total drag. The significant
20 increase in the drag coefficient, combined with a decrease in lift, leads to a severe reduction of the
21 aerodynamic performance of the airfoil.

22

23 **4.2 Effects of Debris Flow concentration and Angle of Attacks on the Erosion Rate of the Blade** 24 **for NACA 63415**

1 This section discusses the sand particles effect on the blade and the rate of erosion caused
2 during the 3-month annually. For Instance, in Egypt Khamasin sandstorm usually occurs between
3 March and May, carrying great quantities of sand and dust from the south into the north Africa [11].
4 The erosion rate variation with the chord length location(x/c) was studied at angles of attack of 2° and
5 10° . The erosion near the trailing edge at Reynolds numbers of 1.6×10^6 , 460×10^3 and 300×10^3 is
6 understood by following the particle trajectory presented in figure 6. This figure show that at higher
7 AOA values, the area of direct contact between pressure side and mean flow increases, which in turn
8 increases the contact with sand particles. However, for smaller AOA values, the sand particles do not
9 come in contact with the trailing edge.

10 Figures from 7 show the erosion rate for a 3-months period of the suction side at angle of
11 attacks 2° and 10° at the three Reynolds number values. As shown in the figure, as the Reynolds
12 number value increases, the amount of erosion through the wind turbine airfoil increase. In addition,
13 the leading edge of the suction side is the most sensitive part where erosion is maximized. The erosion
14 rate could reach up to 0.6 kg/m^2 and 0.23 kg/m^2 in the high sandstorms concentration at Reynolds
15 number values of 1.6×10^6 and 460×10^3 respectively. However, in the case of low sand storms
16 concentration, erosion rate could be 0.07 kg/m^2 , 0.06 kg/m^2 and 0.04 kg/m^2 at Reynolds number values
17 of 1.6×10^6 , 460×10^3 and 300×10^3 respectively.

18 It is observed that the erosion extends more towards the trailing edge at lower values of the
19 Reynolds number and lower angles of attack. For Example, the erosion reached the chord length
20 location(x/c) at 0.04 m at Reynolds number of 1.6×10^6 and angle of attacks 2° . While, by decreasing
21 the Reynolds number value to 300×10^3 , the erosion extended to the chord length location(x/c) of 0.14
22 m at the same angle of attack value. On the other hand, at angle of attack 2° , the erosion almost reached
23 the chord length location(x/c) at 0.04 m, 0.11 m and 0.14 m at Reynold number values of 1.6×10^6 ,
24 460×10^3 and 300×10^3 respectively. while, at angle of attack 10° , the erosion was very limited. This

1 means that the suction side is less affected by the erosion caused due to the sand particles at the higher
2 angle of attack.

3 In contrast, figures from 8 to 13 represent the erosion rate of the pressure side at angle of attacks
4 2° and 10° . It is observed that, the erosion rate is maximized at higher angle of attacks. At angle of
5 attack 10° , the maximum erosion rate was almost 0.46 kg/m^2 and 0.3 kg/m^2 at Reynolds number values
6 of 1.6×10^6 and 460×10^3 respectively in the high sandstorm concentration. While, in case of low
7 sandstorm concentration, the maximum erosion rate at the same angle of attack value was 0.08 kg/m^2 ,
8 0.04 kg/m^2 and 0.06 kg/m^2 at Reynolds number values of 1.6×10^6 , 460×10^3 and 300×10^3
9 respectively.

10 Compared to the suction side, the pressure side has an opposite eroding behavior with the
11 change of the angle of attack values. As the angle of attack increases, the pressure side is more affected
12 by the erosion caused due to the impact of sand particles. As shown in the figures, the erosion was
13 diffused all over the pressure side chord at AOA 10° especially at Reynolds number values of $460 \times$
14 10^3 and 300×10^3 . Thus, this diffusion is significant at low Reynolds number values.

15

16

17 **4.3 Power Curve Output Using BEM Model**

18 The performance of the blade is directly related to its power coefficient (C_p). Therefore, to
19 select an airfoil, the power coefficient of each blade should be found at the operational Reynolds
20 number.

21 Figure 14 (a) shows the C_p of the wind turbine rotor versus the tip speed ratio (TSR) in the
22 clean and rough conditions due to debris flow. A maximum C_p of approximately 0.38 occurs at TSR
23 of 9. Note that, because of the extrapolation used in the BEM model, the accuracy decreases with
24 decreasing TSR. For $\text{TSR} < 7$, the data should be regarded as qualitative. For the clean condition case,
25 the wind turbine rotor can work very well for a TSR from 7.5 to 9.5 since the wind turbine blade nearly

1 maintains the required C_p value. However, as the debris mass flow rate increases, the maximum C_p
2 value obtained decreases. For the case of low debris flow, the maximum C_p value obtained is 0.34,
3 which shows a 11% decrease when compared to clean operation condition. In addition, the maximum
4 C_p value obtained, for the case of high debris flow, is almost 0.28. This is a 26% decrease when
5 compared to the clean operation condition. Figure 14 (b) presents the simulated torque coefficient (C_m)
6 produced for the clean rotor surface as well as rough surfaces for various sand/air volumetric ratios.
7 The maximum torque coefficient produced when wind turbine rotor operates in the clean condition is
8 0.05. This value could be decreased into 0.037 in case of high debris flow.

9 Figure 14 (c) shows the simulated power curves for the clean rotor surface as well as rough
10 surfaces for various sand/air volumetric ratios. This power output was optimized by changing the tip
11 speed ratio value to obtain the optimum value of Torque produced as presented in figure 14 (b).
12 According to figure 14 (c) and the results of this investigation, surface roughness caused by debris can
13 results in a high-energy yield deficit. For the case studied, the decrease in power is predicted to be 10%
14 for low debris flow but could reach 30% for high debris flow compared to clean air.

15

16 **4.4 Nordtank NTK 500/41 Wind Turbine Case Study**

17 Large-scale Wind turbines (LSWT) with a rated power of 50kW-1 MW are a mature
18 technology and should experience rapid growth in coming decades [5]. Nordtank NTK 500/41 is an
19 example of a large-scale commercial wind turbine which produces a rated power of 500 kW. This wind
20 turbine was installed in 1992 for testing and its performance was investigated extensively during 1992-
21 1999 [53]. The NTK 500/41 is a stall regulated (fixed pitch) turbine with fixed rotational speed control
22 strategy. Table 6 shows the main parameters of Nordtank NTK 500/41 [53].

23 Using QBlade software and the above rotor simulation curves, the wind turbine power curve
24 produced by the Nordtank NTK 500/41 wind turbine was simulated for wind speeds 4-25 m/s. The
25 mechanical power available from the blades shown in Fig. 14 (c) was used by the QBlade simulation

1 for operating the wind turbine at its optimal tip speed ratio for maximum electrical power generation.
2 When the electrical generation load is applied, the blades will rotate at a slower speed than the stall
3 limited rotational speed, and the electrical power output will therefore be reduced compared to the
4 mechanical power from the blades shown in Fig. 14 (c). Figure 14 (d) presents Nordtank NTK 500/41
5 power curve at different wind speeds in clean and rough conditions. The power curve is oscillating at
6 the end of it due to small errors in the iterative calculation, which are magnified in the power output
7 for high wind speeds. Under sandstorm conditions, the power losses from debris flow seen in Figure
8 14 (d) would directly affect the turbine power. For the current case study, the maximum power loss is
9 almost 8 %, 14 % and 22 % in low, medium and high debris flow concentrations, respectively.

10

11 **4.5 Neural Network Erosion Per Unit Area Prediction**

12 After the training process of the proposed ANN has been done, the eight excluded patterns
13 were used to test the reliability of this developed ANN. The ANN and CFD outputs were compared
14 and the absolute percentage error (APE) for each pattern was calculated. Figure 15 (a) shows a
15 comparison between the CFD and ANN output for the total erosion on chord per unit area for each
16 pattern used in the reliability test for the developed neural network. Figure 15 (b) APE between the
17 CFD and ANN output for the total erosion on chord per unit area for each pattern used in the reliability
18 test for the developed neural network. As seen in the figure, the average APE is 9.42 %, the maximum
19 APE is 14.58 % and the minimum APE is 4.33 %. Figure 15 (c) shows the APE between the CFD and
20 ANN output for the total erosion on chord per unit area for all the patterns used in the ANN training
21 progress and the reliability test. It has also been listed in the last column in table 4. According to [54],
22 the average APE is used to evaluate the approximation performance precision of the ANN models.
23 The average APE of this constructed metamodel is 4.91 %. This value is normally considered
24 acceptable, and should be given particular consideration since, according to the literature survey, this
25 is the first ANN model which could estimate the HAWT rate of erosion. Increasing the patterns of the

1 input and output data in the training process helps to maximize the approximation accuracy of this
2 neural network model. However, this increased training will increase the computational cost and
3 potentially reduce the applicability of the model to other new scenarios.

4 **5.0 Conclusion**

5 In this work, the effect of debris/air flow on the aerodynamic performance of horizontal axis
6 wind turbines (HAWT) was investigated using CFD-BEM modelling. Before evaluating the rotor
7 performance using BEM theory, lift and drag coefficients were obtained as a function of angle of attack
8 through 2D CFD simulations. The CFD simulation results were validated using experimental data in
9 the clean condition. Then the lift and drag coefficient values during debris flow simulations were
10 obtained from CFD and used in the BEM theory. Large-scale Wind turbines case study was presented.
11 Power curves for the wind turbine rotor were obtained and estimated in different debris flow values.
12 Results based on the BEM method showed that the power generated under sandstorm conditions can
13 decrease 30% compared to normal conditions. An artificial neural network has been proposed to
14 predict the total amount of erosion occurred per unit area as a function of the operating conditions of
15 the wind turbine blade, which are: the debris flow rate, the Reynolds number and the angle of attack.
16 Therefore, researchers working on wind turbine design, optimization, diagnosis and maintenance
17 should be aware of the debris flow issues.

18

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21 Transport.

22

23 **Data Availability**

24 The data that supports the findings of this study are available within the article.

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Table 1 The equivalent values of the characteristic ratio between the sand and air (α) for the three mass flow rate values at Reynolds number 1.6 million.

Mass Flow Rate (kg/s)	Characteristic Ratio Between the Sand and Air (α)
100	0.4×10^{-4}
200	0.9×10^{-4}
400	1.7×10^{-4}

Table 2 The equivalent values of the characteristic ratio between the sand and air (α) for the three mass flow rate values at Reynolds number 460,000.

Mass Flow Rate (kg/s)	Characteristic Ratio Between the Sand and Air (α)
100	1.5×10^{-4}
200	3.0×10^{-4}
400	5.9×10^{-4}

Table 3 The equivalent values of the characteristic ratio between the sand and air (α) for the three mass flow rate values at Reynolds number 300,000.

Mass Flow Rate (kg/s)	Characteristic Ratio Between the Sand and Air (α)
100	2.3×10^{-4}
200	4.5×10^{-4}

Table 4 Input and output patterns for CFD and neural network

Pattern Number	Input Data			CFD Output Data	ANN Output Data	ANN Percentage Error – Absolute Value(%)
	Debris Flow Rate(kg/s)	Reynolds Number	AOA	Total Erosion on Chord (g/m ²)	Total Erosion on Chord (g/m ²)	
1	100	300000	0	579	613	5.95
2	100	300000	2	654	705	7.79
3	100	300000	4	695	664	4.43
4	100	300000	6	620	710	14.53
5	100	300000	8	1069	993	7.07
6	100	300000	10	1008	1024	1.61
7	200	300000	0	1387	1356	2.19
8	200	300000	2	1266	1139	10.02
9	200	300000	4	1000	983	1.70
10	200	300000	6	1475	1543	4.66
11	200	300000	8	2719	2536	6.72
12	200	300000	10	1944	1945	0.07

13	100	460000	2	790	709	10.25
14	100	460000	4	655	684	4.50
15	100	460000	6	797	694	12.98
16	100	460000	8	855	905	5.91
17	100	460000	10	1140	1021	10.50
18	200	460000	0	1538	1315	14.47
19	200	460000	2	1306	1349	3.28
20	200	460000	4	965	1025	6.18
21	200	460000	6	1321	1211	8.28
22	200	460000	8	1866	2029	8.73
23	200	460000	10	2282	2284	0.08
24	400	460000	0	2457	2455	0.08
25	400	460000	2	1997	2013	0.81
26	400	460000	4	1759	1751	0.45
27	400	460000	6	2255	2061	8.62
28	400	460000	8	2926	2636	9.91
29	400	460000	10	3645	3610	0.97
30	100	1600000	0	770	819	6.40
31	100	1600000	2	575	541	6.01
32	100	1600000	4	389	342	12.00
33	100	1600000	6	655	657	0.34
34	100	1600000	8	832	868	4.31
35	100	1600000	10	832	801	3.75
36	200	1600000	0	1539	1526	0.85
37	200	1600000	2	1151	1001	12.97
38	200	1600000	4	890	891	0.16
39	200	1600000	6	1234	1210	1.98
40	200	1600000	8	1663	1686	1.41
41	200	1600000	10	1554	1501	3.39
42	400	1600000	0	3078	3076	0.05
43	400	1600000	2	2457	2456	0.03
44	400	1600000	4	1554	1621	4.30
45	400	1600000	6	2621	2619	0.07
46	400	1600000	8	3808	3807	0.03
47	400	1600000	10	3107	3102	0.18

Table 5 The Excluded input and output patterns from the neural network training progress

Testing Pattern Number	Input Data			CFD Output Data	ANN Output Data	ANN Percentage Error (%) – Absolute Value
	Debris Flow Rate (kg/s)	Reynolds Number	AOA	Total Erosion on Chord (g/m ²)	Total Erosion on Chord (g/m ²)	
4	100	300000	6	620	710.3860241	14.58
8	200	300000	2	1266	1139.177416	10.02
14	100	460000	4	655	684.0733472	4.44
21	200	460000	6	1321	1211.315698	8.30
27	400	460000	6	2255	2060.659089	8.62
32	100	1600000	4	389	341.9535792	12.09
37	200	1600000	2	1151	1001.485368	12.99
44	400	1600000	4	1554	1621.22193	4.33

Table 6: The main parameters of Nordtank NTK 500/41

Rotational Speed	27.1 rpm
Rotor Radius	20.5 m
Number of Blades	3
Cut-in Wind Speed	4 m/s
Cut-out Wind Speed	25 m/s
Profile	NACA 63-4xx

Figure Captions

Figure 1. shows the layout of the wind turbine blade and the airfoil used

Figure 2. Validation of lift (C_l) and drag (C_d) coefficients for the current study: (a) lift coefficient for NACA 63415 at Reynolds Number 1.6×10^6 (b) drag coefficient at Reynolds Number for NACA 63415 1.6×10^6 (c) lift coefficient for NACA 63415 at Reynolds Number 460×10^3 (d) drag coefficient for NACA 63415 at Reynolds Number 460×10^3 (e) lift coefficient for NACA 63415 at Reynolds Number 300×10^3 (f) drag coefficient for NACA 63415 at Reynolds Number 300×10^3 . Experimental results from Bak et al. [25] all for $Re = 1.6 \times 10^6$ and shown in all graphs for comparison.

Figure 3. Velocities and forces related to the wind turbine blade

Figure 4. (a) Implemented Neural Network in MATLAB software (b) The neural network outputs with respect to targets (the correct CFD output for each of the input data) for training, validation, and test sets

Figure 5. lift (C_l) and drag (C_d) coefficients Variation with angle of attacks in clean and sandy conditions. (a) lift coefficient at Reynolds Number 1.6×10^6 (b) lift coefficient at Reynolds Number 460×10^3 (c) lift coefficient at Reynolds Number 300×10^3 (d) drag coefficient at Reynolds Number 1.6×10^6 (e) drag coefficient at Reynolds Number 460×10^3 (f) drag

Figure 6. Particle traces colored by particle velocity magnitude at angle of attack 2° and 10° (a) Reynolds number 1.6×10^6 (b) Reynolds number 460×10^3 (c) Reynolds number 300×10^3

Figure 7. Suction side erosion (a) Erosion at Reynolds number 1.6×10^6 and AOA 2° (b) Erosion at Reynolds number 1.6×10^6 and AOA 10° (c) Erosion at Reynolds number 460×10^3 and AOA 2° (d) Erosion at Reynolds number 460×10^3 and AOA 10° (e) Erosion at Reynolds number 300×10^3 and AOA 2° (f) Erosion at Reynolds number 300×10^3 and AOA 10° . Only selected regions are shown for clarity. For locations not shown within the range in any figure, the erosion is zero.

Figure 8. Pressure side erosion rate for 3-months period annually at Reynolds number of 1.6×10^6 and AOA of 2° for NACA 63415

Figure 9. Pressure side erosion rate for 3-months period annually at Reynolds number of 1.6×10^6 and AOA of 10° for NACA 63415

Figure 10. Pressure side erosion rate for 3-months period annually at Reynolds number of 460×10^3 and AOA of 2° for NACA 63415

Figure 11. Pressure side erosion rate for 3-months period annually at Reynolds number of 460×10^3 and AOA of 10° for NACA 63415 (a) Chord range (x/c) from 0 to 0.5 (a) Chord range (x/c) from 0 to 0.5

Figure 12. Pressure side erosion rate for 3-months period annually at Reynolds number of 300×10^3 and AOA of 2° for NACA 63415

Figure 13. Pressure side erosion rate for 3-months period annually at Reynolds number of 300×10^3 and AOA of 10° for NACA 63415 (a) Chord range (x/c) from 0 to 0.5 (a) Chord range (x/c) from 0 to 0.5

Figure 14. (a) Power Coefficient Vs TSR using BEM Model in clean and sandy conditions (b) Torque Coefficient Vs TSR using BEM Model in clean and sandy conditions (c) Power Output using BEM Model in clean and sandy conditions (d) Nordtank NTK 500/41 power curve at different wind speeds in clean and rough conditions

Figure 15. (a) Comparison between the CFD and neural network output values for the total erosion on chord per unit area (b) The absolute percentage error between the CFD and neural network output values for the total erosion on chord per unit area for each pattern used in the reliability test (c) The absolute percentage error between the CFD and neural network output values for the total erosion on chord per unit area for the whole patterns used in neural network training process and reliability test.