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Fluid interfaces with very sharp tips in viscous flow

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1 When a fluid interface is subjected to a strong viscous flow, it tends
 2 to develop near-conical ends with pointed tips so sharp, their radius
 3 of curvature is undetectable. In microfluidic applications, tips can
 4 be made to eject fine jets, from which micron-sized drops can be
 5 produced. Here we show theoretically that the opening angle of the
 6 conical interface varies on a logarithmic scale as function of the dis-
 7 tance from the tip, owing to non-local coupling between the tip and
 8 the external flow. Using this insight we are able to show that the tip
 9 curvature grows like the exponential of the square of the strength
 10 of the external flow, and to calculate the universal shape of the in-
 11 terface near the tip. Our experiments confirm the scaling of the tip
 12 curvature as well as of the interface's universal shape. Our analytical
 13 technique, based on an integral over the surface, may also have far
 14 wider applications, for example treating problems with electric fields,
 15 such as electrospays.

Free surface flows | singularities | selective withdrawal | microfluidics

1 In many problems of science and engineering, or in daily
 2 life, one is confronted with fluid interfaces subject to strong
 3 external flows. For example, consider a bubble rising in a
 4 viscous fluid (4) (such as in a shampoo bottle), emulsions of
 5 drops or bubbles being stirred (2, 5), a viscous layer being
 6 withdrawn from near an interface (6), or two fluids meeting in
 7 a microfluidic channel (7).

8 In Fig. 1 we show three typical situations: at the top,
 9 a viscous liquid (light) flows out through a circular hole at
 10 the bottom of the picture, deforming the interface between
 11 liquid and air - a flow geometry known as selective withdrawal
 12 (1, 8, 9). The interface is focused into a near-conical shape,
 13 which ends in a tip so small, it can no longer be resolved
 14 by optical means. Therefore, in the inset we show profiles
 15 obtained from a numerical simulation: as the tip is plotted
 16 with increasing resolution (decreasing a), the observed opening
 17 angle increases. This is a reflection of the logarithmic variation
 18 of the interface slope, which is the key feature of the solution
 19 described below.

20 In the middle we see a drop of liquid, whose viscosity
 21 is much smaller than that of the outer liquid, being drawn
 22 apart by an extensional flow. Once more, very sharp tips
 23 are formed at the ends of the drop, and the ends appear
 24 conical. At the bottom, a conical interface is produced when
 25 a stream of oil is forced by water in a microfluidic assembly
 26 (7, 10, 11). However, now a liquid thread escapes from the tip
 27 (a phenomenon called tipstreaming (12)), whose subsequent
 28 decay is a means of producing micron-sized droplets in a highly
 29 reproducible fashion (3) of interest for chemical analysis and
 30 in soft matter research.

31 Similar conical structures, known as Taylor cones, appear
 32 when a liquid is placed in a strong electric field (13–15). How-
 33 ever, the Taylor cone is generally unstable, and a tiny jet
 34 is emitted from its tip (12, 16–18), akin to the tipstreaming
 35 phenomenon described above. This has led to a vast number

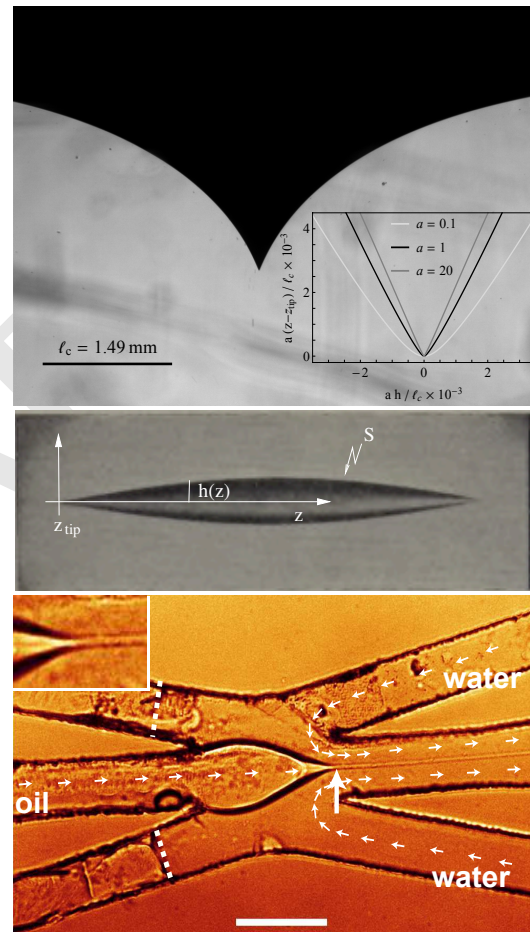


Fig. 1. Top: A liquid-air interface is deformed into a sharp tip as the fluid below escapes through a hole at the bottom (1); ℓ_c is the capillary length. The dimensionless source strength (flow rate q) is $\chi = q\eta/(\ell_c^2\gamma) = 6.31$, where η the dynamic viscosity, and γ the surface tension. The local capillary number (details below) is $Ca_{\text{tip}} = 1.64$. The inset shows closeups of the tip found from a numerical simulation of the experiment: as one zooms in, the opening angle increases. Middle: A low-viscosity drop in an extensional flow generated by four rollers (2), with capillary number $Ca = GR_d\eta/\gamma = 0.7$, and unperturbed drop radius $R_d = 0.25 \text{ cm}$, where G is the extension rate of the flow. Bottom: A stream of oil is being forced into a sharp tip by water flowing in the reverse direction at a junction of small channels (3). A jet a few microns thick is ejected from the tip (see inset). The scale bar represents $75 \mu\text{m}$.

of applications in spraying and materials processing (19, 20), but the fundamentals of how a Taylor cone leads to secondary structures has not been understood properly.

Almost a century ago, G.I. Taylor set up a research program to understand the production and stirring of emulsions (2, 21), by considering drops subjected to simple external forcing, as seen in Fig. 1. After stirring, very small drops of about 1/100 the size of the original drop can be observed (2), which can be attributed to the ejection of small threads, similar to those shown at the bottom of the figure. Taylor later developed a theory for the shape of slender drops of small or negligible viscosity (22), which predicted conical ends with tips of vanishing size, violating the assumption of smoothness, on which continuum theory is based (23).

Later refinements of Taylor's theory (24) show that it breaks down near the tips. Other theoretical proposals for drop shapes also exhibit singular tips (24, 25). Our later numerical calculations (26) suggest a finite curvature at the tips, which grows exponentially with the square of the external flow speed. It remains to understand this issue of singular tips theoretically, and to calculate the true shape near the ends. This task has hitherto proven impossible, since it represents a fully non-linear, three-dimensional free-surface problem, for which few analytical methods of solution exist. In particular, a slender-body or lubrication type approach fails here, since the end is not a slender shape.

The axisymmetric geometry we consider is particularly significant, since it represents "optimal" focusing of the flow. In the two-dimensional analogue of the same problem, the interface shape is a cusp (27, 28), whose tip traces out a line in three dimensions. Since this involves focusing along a whole line, it is much less efficient than focusing on a single point, as in the present problem. As a result, the curvature only increases exponentially with the flow strength in the quasi two-dimensional case, instead of the exponential of the square. Our analytical calculation of the curvature of an axisymmetric tip establishes theoretically that steady flows with surface tension are always smooth, although rounding may occur on very small scales only.

The idea of our analysis is that the region near the tip is on a scale very different from the bulk of the flow. Thus if we introduce the distance $\zeta = (z - z_{\text{tip}})\kappa_m$ from the tip (see Fig. 1, middle), made dimensionless with (twice) the mean

curvature of the tip κ_m , the radius $h(z)$ of an axisymmetric interface can be written in the form

$$h(z) = \kappa_m^{-1} H(\zeta), \quad [1]$$

where $H(\zeta)$ should be a universal similarity solution, independent of the geometry or of the external flow. In (26) we provided numerical evidence for Eq. (1), comparing $H(\zeta)$ calculated from a drop in extensional flow, and from a selective withdrawal configuration. Below we calculate the similarity function $H(\zeta)$ analytically for large ζ , which corresponds to the limit of large κ_m at a small but constant distance from the tip. We then use this to calculate the scaling of the tip curvature as function of the local flow strength.

Asymptotic analysis near the tip. To calculate the profile and the flow near the ends, we consider the simplest case of axisymmetric viscous (Stokes) flow, with vanishing viscosity inside. Even if inertia becomes important on a large scale, this will still be a correct description locally, where the local Reynolds number is small, i.e. viscous forces are much stronger than inertial forces. An axisymmetric cross section is a good approximation even if the external flow is not axisymmetric (29, 30), for example in a shear flow. A finite viscosity fluid inside the tip eventually leads to a tipstreaming bifurcation (22, 31), but we will be describing the regime before this occurs, yet tips have become very sharp.

Instead of solving the flow equations in the bulk, with boundary conditions applied at the free surface, we use the equivalent boundary integral description (32), which is formulated on the free surface alone, and which has proven extremely effective addressing free surface problems numerically with very high resolution (26, 32, 33). However, this technique is very unwieldy as an analytical tool, since the surface to be integrated over is unknown a priori. Here we break new ground by using the boundary integral technique to find analytical solutions, using the fact that the interface slope is changing very slowly.

The integral equation to be solved for the velocity $\mathbf{v}(\mathbf{x}_1)$ on the surface S (see Fig. 1, middle, more details in *Materials and Methods*) is (32)

$$\frac{\mathbf{v}(\mathbf{x}_1)}{2} = \mathbf{v}^{(ext)}(\mathbf{x}_1) - \int_S \kappa(\mathbf{x}_2) \mathbf{J} \cdot \mathbf{n} d\sigma_2 - \int_S \mathbf{v}(\mathbf{x}_2) \cdot \mathbf{K} \cdot \mathbf{n} d\sigma_2. \quad [2]$$

Velocities are written in units of the capillary speed $v_c \equiv \gamma/\eta$, with γ the surface tension and η the dynamic viscosity. In Eq. (2), $\mathbf{v}^{(ext)}(\mathbf{x}_1)$ is an externally imposed velocity, κ the mean curvature, and \mathbf{n} the outward normal. The kernel $\mathbf{J} \cdot \mathbf{n}$ is the velocity at \mathbf{x}_1 , generated by a point force at \mathbf{x}_2 in Stokes flow, and $\mathbf{K} \cdot \mathbf{n}$ is the corresponding stress tensor at \mathbf{x}_1 .

Since the flow is axisymmetric, one can perform the azimuthal integration explicitly (32), and choose the dimensionless distance ζ from the tip as the integration variable. Transforming to the logarithmic variable $l = \ln \zeta$, we obtain from Eq. (2)

$$\frac{\mathbf{v}(l_1)}{2} = v_{\text{tip}} \mathbf{e}_z + \mathbf{v}^{(J)} - \int_{-\infty}^{\ln(L\kappa_m)} \mathbf{v}(l_2) \zeta_2 \mathbf{k}(l_1, l_2) dl_2, [3]$$

$$\mathbf{v}^{(J)}(l_1) = - \int_{-\infty}^{\ln(L\kappa_m)} \zeta_2 (\kappa(l_2)/\kappa_m) \mathbf{j}(l_1, l_2) dl_2, \quad [4]$$

Significance Statement

Turn a shampoo bottle upside down, and the rising bubble develops a very sharp tip at its rear. Similar conical structures have been observed placing a drop in an electrical field. Owing to the significance of such structures in chemical processing or in geophysics, they have been studied for almost a century. Here we present a consistent analysis of the shape near the tip in flow. We show that the size of the tip always remains finite, but decreases so rapidly in size with the flow strength, that it may be considered zero for practical purposes.

S. C. du P. performed selective withdrawal experiments; J. E. developed the theory and wrote the manuscript

The authors have no conflict of interest to declare

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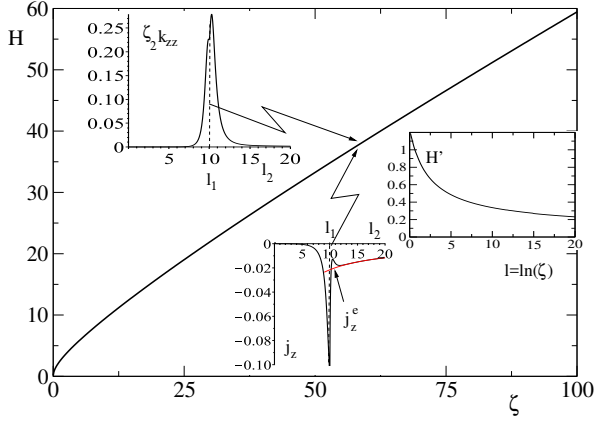


Fig. 2. Schematic representation of our treatment of Eq. (6). The self-similar profile $H(\zeta)$ has a slope H' which changes on a logarithmic scale, see inset. Integrands like $\zeta_2 k_{zz}$ are peaked at $l_2 = l_1$, except j_z , which decays like j_z^e for large l_2 .

to $v_z(l_1)$, multiplied by the area of the peak. The exception is the integral over j_z , which decays very slowly for large l_2 .

Namely, taking a cone of slope s_2 as the interface, one finds that for $\Delta \rightarrow \infty$

$$j_z(l_1, l_2) \rightarrow j_z^e \equiv -\frac{s_2^2(1-s_2^2)}{4(1+s_2^2)^{3/2}} \approx -\frac{s_2^2}{4}, \quad [7]$$

where j_z^e is shown as the red line in the plot of j_z in Fig. 2. In the same limit of small slopes, $\zeta_2 \kappa / \kappa_m \approx 1/s_2$; thus the integrand of $v_z^{(J)}$ in Eq. (4) is $\zeta_2(\kappa/\kappa_m)j_z \approx -s_2/4$. On the other hand, for $\Delta \rightarrow -\infty$, $j_z(l_1, l_2)$ vanishes rapidly (cf. Fig. 2), and we obtain

$$v_z^{(J)} \approx \frac{1}{4} \int_{l_1}^{\ln(L\kappa_m)} s(l_2) dl_2. \quad [8]$$

Clearly since $s \propto 1/\sqrt{l_2}$, this integral makes a contribution $\propto \sqrt{\ln(L\kappa_m)}$ from its upper limit. Once this has been removed by balancing it against v_{tip} in Eq. (6), we anticipate that $v_z \propto \sqrt{l_1}$ from the lower limit.

Now we compute the other integrals in the same spirit as integrals over cones, but using the fact that the remaining kernels are localized about $\Delta \approx 0$, and decay for large Δ . Thus since $\mathbf{j}(l_1, l_2) = \mathbf{j}(s_1, \Delta)$ for a conical interface, we can approximate

$$v_r^{(J)} \approx (\zeta_1 \kappa(l_1) / \kappa_m) \int_{-\infty}^{\infty} j_r(s_1, \Delta) d\Delta = 0, \quad [9]$$

which is confirmed by an expansion of $j_r(s_1, \Delta)$ for small s_1 .

The kernels $\mathbf{k}(l_1, l_2) \approx \mathbf{k}(s_1, \Delta)$ can be treated in the same way. Beginning with z -component of Eq. (6),

$$K_1 \equiv \int_{-\infty}^{L\kappa_m} \zeta_2 k_{zz} v_z(l_2) dl_2 \approx v_z(l_1) \int_{-\infty}^{\infty} \zeta_2 k_{zz}(s_1, \Delta) d\Delta, \quad [10]$$

where we have used, as illustrated in Fig. 2, that $\zeta_2 k_{zz}$ is strongly peaked, so that $v_z(l_2)$ varies little over width of the peak. The remaining integral (the area of the peak) can be shown to be 1/2 to leading order in an expansion for small s_1 (see *Materials and Methods*); thus $K_1 \approx v_z(l_1)/2$. In the same vein,

$$K_3 \equiv \int_{-\infty}^{L\kappa_m} \zeta_2 k_{zr} v_r(l_2) dl_2 \approx v_r(l_1) \int_{-\infty}^{\infty} \zeta_2 k_{zr}(s_1, \Delta) d\Delta. \quad [11]$$

Now the integral is $\approx s_1/2$ to leading order, which becomes small, and K_3 can be neglected compared to K_1 .

Coming to the r -component of Eq. (6), to a first approximation we have

$$K_4 \equiv \int_{-\infty}^{L\kappa_m} \zeta_2 k_{rr} v_r(l_2) dl_2 \approx v_r(l_1) \int_{-\infty}^{\infty} \zeta_2 k_{rr}(s_1, \Delta) d\Delta. \quad [12]$$

To leading order, the value of the last integral is $-1/2$, so that $K_4 \approx -v_r/2$; but this cancels $v_r/2$ on the left, and we have to go to the next order, which yields $K_4 \approx v_r(l_1)(-1/2 + s_1^2/4)$. The remaining component is

$$K_2 \equiv \int_{-\infty}^{L\kappa_m} \zeta_2 k_{rz} v_z(l_2) dl_2 \approx v_z(l_1) \int_{-\infty}^{\infty} \zeta_2 k_{rz}(s_1, \Delta) d\Delta, \quad [13]$$

but the integral is again zero to leading order, and even the next order vanishes. To capture the leading nonzero contribution,

where L is a characteristic size of the setup, such as a drop size, and $v_{\text{tip}} = v_z^{(\text{ext})}(z_{\text{tip}})$ is the external velocity at the tip. The kernels \mathbf{j} and \mathbf{k} correspond to $\mathbf{J} \cdot \mathbf{n}$ and $\mathbf{K} \cdot \mathbf{n}$ in Eq. (2), integrated over θ , and written in units of the tip curvature κ_m .

The integrands in Eq. (3) and Eq. (4) are now invariant under a scale transformation: if the surface were a perfect cone of slope s , the integrands would be functions of s and $\Delta = l_2 - l_1$ alone. Thus if all integrals were convergent in the limit $L\kappa_m \rightarrow \infty$, each term in Eq. (3) would be a function of s alone, and a cone would result as a (similarity) solution to the flow equations. However, as first pointed by Taylor (22), and confirmed by Buckmaster (24), such a conical solution does not exist. Indeed, we will see that the z -component of $\mathbf{v}^{(J)}$ in Eq. (4) is in fact divergent for $L\kappa_m \rightarrow \infty$, so L cannot be eliminated from the problem. Instead, we must keep the upper limit in Eq. (4) finite, and cancel the resulting contribution against the external velocity v_{tip} , introducing an intrinsic coupling between the external flow and the local behavior at the tip.

The key idea of our approach is to suppose that the interface has the form (cf. Fig. 2):

$$H(\zeta) = \zeta s(\ln \zeta) \equiv \zeta s(l), \quad [14]$$

where $s(l)$ is a local slope, which varies on a logarithmic scale. We will show below that for large l (far from the tip), which is the asymptotic behavior which interests us, $s(l) \propto 1/\sqrt{l}$, i.e. the slope becomes small, as shown in the inset of Fig. 2. In doing so, we are always in the limit $\zeta \ll L\kappa_m$, and hence the inner solution near the end applies. If we write Eq. (3) in the form

$$\frac{1}{2} \begin{pmatrix} v_z \\ v_r \end{pmatrix} = \begin{pmatrix} v_{\text{tip}} + v_z^{(J)} - K_1 - K_3 \\ v_r^{(J)} - K_2 - K_4 \end{pmatrix}, \quad [15]$$

in evaluating the dominant contribution to each term we can assume the integral to be over a cone of constant slope (since $s(l)$ is varying very slowly), which is a tractable problem.

Our treatment of Eq. (6) is illustrated in Fig. 2: to find $v_{z/r}$ at a point $l_1 = \ln(\zeta_1)$, according to Eq. (3) and Eq. (4), we have to perform an integral over l_2 . However, as shown for the example of $\zeta_2 k_{zz}$, integrands are peaked at $l_2 = l_1$, and hence to leading order the contribution to K_1 is proportional

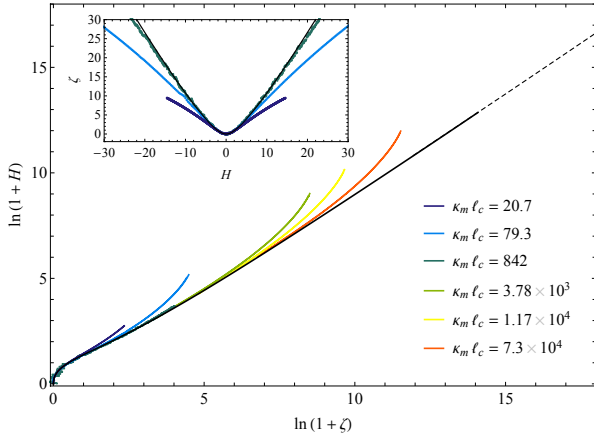


Fig. 3. Comparison between the similarity solution $H(\zeta)$ as predicted by theory (black solid line), and six different experimental data sets for different values of $\kappa_m \ell_c$, rescaled according to Eq. (1) (colored lines); $\ell_c = \sqrt{\gamma/(\rho g)} = 1.49\text{mm}$ is the capillary length. The main panel shows all six experimental data sets on a logarithmic scale, the inset only the first three sets on a linear scale. The dashed line is the asymptotic behavior Eq. (9) with $l_0 = 2.55$. For the first three experimental profiles, κ_m has been measured directly, (colored dots in Fig. 4), for the profiles with the highest tip curvatures, κ_m has been calculated from a numerical simulation (colored pluses in Fig. 4).

(solid line), using a full numerical simulation of a drop in an extensional flow (26); the constant of integration l_0 was adjusted to match with the numerical profile. The major conclusion from Eq. (10) is that the pointed ends seen for example in Fig. 1 only appear conical since s is nearly constant over an observable range of scales, but this opening angle will change when the scale of observation is changed, and one zooms into the tip.

The theoretical tip similarity solution $H(\zeta)$ (black solid line) is compared in Fig. 3 to six experimental data sets from a selective withdrawal experiment, described in detail in (1) (colored lines). A container 3 cm wide is half filled with silicone oil of viscosity $\eta = 30\text{ Pa s}$ and surface tension $\gamma = 2.13 \times 10^{-2}\text{ Nm}^{-1}$. The liquid is evacuated through a circular sink hole of 1 mm diameter in a solid plate, at a constant flow rate of $q = 9.97 \times 10^{-9}\text{ m}^3/\text{s}$, but replenished at a rate which is slightly smaller, so that the distance between the hole and the mean liquid level decreases adiabatically, over a period of several hours.

As the liquid-air interface comes closer to the sink hole, it is increasingly deformed. We record the distance z_t between interface tip and the hole, as well as the shape of the interface. The experimental cell is lit from behind, so that light is refracted away by the interface, whose cross section appears black (cf. Fig. 1), top. The measured interface shape (colored lines) has been rescaled according to Eq. (1), with tip curvatures given in the figure. The crossover between the similarity solution $H(\zeta)$ and the outer solution, which is shaped by the sink flow out of the container, takes place at a fixed outer scale $z - z_{\text{tip}}$. Thus to increase the range of the similarity variable ζ over which to compare to the experiment, one has to increase the tip curvature.

For the first three profiles of Fig. 3, the tip curvature κ_m has been measured directly by interpolating the tip region. The teal curve with the highest curvature shows significant noise, as it has been zoomed in to the limit of our resolution. To be able to compare to theory over a larger range, we included three profiles with a much larger tip curvature, focusing on the far field. The tip sizes of these profiles have become so small, that they are below our optical resolution. Instead, we calculate κ_m from a full numerical simulation of the flow equations, which matches closely the experimental geometry. We can argue on the basis of Fig. 4 below that the numerical estimates of the curvature are very reliable.

To compute the tip curvature κ_m analytically, we insert Eq. (10) into the First of Eq. (9) to find to leading order

$$-\frac{\sqrt{l}}{2} \approx -\text{Ca}_{\text{tip}} + \frac{1}{4} \int_l^{\ln(L\kappa_m)} \frac{dl_2}{\sqrt{l_2}}, \quad (10)$$

where the local capillary number $\text{Ca}_{\text{tip}} = -v_{\text{tip}}$ is the dimensionless z -velocity at the tip. From this we find $\text{Ca}_{\text{tip}} \approx \sqrt{\ln(L\kappa_m)}/2$, and thus

$$\kappa_m \propto L^{-1} \exp(4\text{Ca}_{\text{tip}}^2), \quad (11)$$

which is the main result of this paper. For Eq. (11) to be valid, Ca_{tip} needs to be sufficiently large so that there is an appreciable range over which s follows the scaling Eq. (10). Owing to the faster-than-exponential growth of κ_m , Eq. (11) shows that while the curvature always remains finite, it can easily reach microscopic dimensions even at moderate values

we expand $v_z(l_2) = v_z(l_1) + v'_z(l_1)\Delta + \dots$, using that k_{rz} is peaked around $\Delta = 0$. Thus an improved approximation becomes

$$K_2 \approx v'_z(l_1) \int_{-\infty}^{\infty} \Delta \zeta_2 k_{rz}(s_1, \Delta) d\Delta \approx -v'_z(l_1) \frac{s_1}{2}$$

to leading order.

This completes the necessary calculations of all the integrals. Next we will interpret Eq. (6) as a dynamical system for $s(l), v_z(l)$ and $v_r(l)$.

Flow equations and scaling. To summarize, Eq. (6) becomes to leading order, putting $l_1 = l \equiv \ln \zeta$,

$$v_z = v_{\text{tip}} + \frac{1}{4} \int_l^{\ln(L\kappa_m)} s(l_2) dl_2, \quad 0 = s v'_z/2 - v_r s^2/4. \quad [9]$$

The contribution v_{tip} of the external flow cancels against the contribution from the upper limit of the $v_z^{(J)}$ integral, as we see below. However, we first calculate $s(l)$ to leading order. To close the system Eq. (9), we use that streamlines must be parallel to the free surface, and thus $v_r/v_z = H' = s + s' \approx s$. Differentiating the First of Eq. (9) with respect to l , we find $v'_z = -s/4$, and substituting into the second equation gives $v_r = -1/2$. This corresponds to the familiar result (34, 35) $-v_c/2$ for the rate of collapse of a cylindrical cavity, remembering that velocities have been made dimensionless with the capillary speed v_c . It follows that $v_z = -1/(2s)$ from the kinematic condition, leading to $s' = -s^3/2$. Solving, we find the asymptotic solution far from the tip (but still in the self-similar region) to be

$$s = (l - l_0)^{-1/2}, \quad v_r = -\frac{1}{2}, \quad v_z = -\frac{(l - l_0)^{1/2}}{2}, \quad [10]$$

consistent with the previous assumption of small slopes for $l \rightarrow \infty$. In Fig. 3, Eq. (10) (dashed line) is found to be in excellent agreement with $H(\zeta)$ as calculated from Eq. (1)

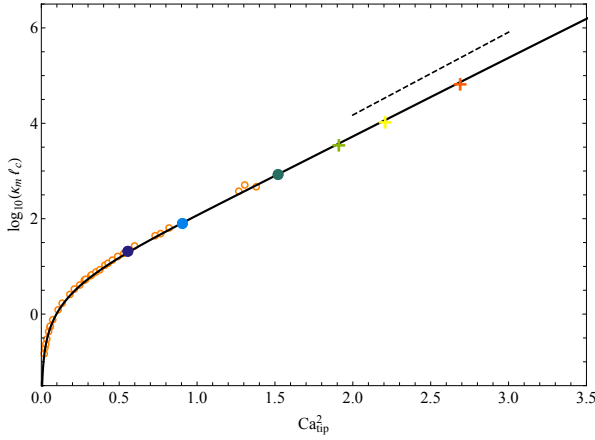


Fig. 4. The dimensionless tip curvature $\kappa_m \ell_c$, as function of the square of the local capillary number Ca_{tip} , based on the *unperturbed* velocity at the tip. The solid line is a numerical simulation (details in (26)), parameters the same as Fig. 1 (top). The open orange circles are the experimental measurements, applying no adjustable parameters. The filled circles correspond to the measured curvature of the first three profiles of Fig. 3, the pluses are the curvatures for the last three profiles, based on the numerical simulation. The dashed line has a slope of $a = 4/\ln 10$, corresponding to the theoretical prediction Eq. (11).

of the capillary number. For example, taking the drop in the middle of Fig. 1, Eq. (11) would predict a radius of curvature of about 10^{-89} m! This shows that while the tip itself is often too small to observe, what is experimentally relevant is the slow variation of the slope described by our asymptotic theory Eq. (10).

In Fig. 4 we test Eq. (11) against experiment (circles) and simulation (solid line). As the capillary number increases, the logarithm of the curvature, plotted as function of the square of the capillary number at the tip, quickly converges toward a straight line. The dashed line has a slope of $4/\ln 10$, which is the asymptotic prediction of Eq. (11), and is seen to agree very well with both simulation and experiment. It can be shown (36) that the slight over prediction of the slope can be traced to a slow variation of the *prefactor* in Eq. (11) on the capillary number.

The orange circles in Fig. 4 come from measurements of the tip curvature in the selective withdrawal geometry for the same parameters as in Fig. 3. The external velocity at the tip was calculated from the flow field of a point source inside a solid wall, evaluated at z_t . The black solid line is the result of a numerical simulation performed using the boundary integral method (26). The simulation assumes a point source in a solid wall, unbounded in the horizontal direction (instead of a finite experimental cell), an approximation which works very well, except far from the center (26). No adjustable parameters were applied to achieve the near-perfect agreement between simulation and experiment. The colored solid circles are also obtained from direct measurements of the tip curvature, and correspond to the first three profiles of Fig. 3. The pluses have been picked out from the solid line, and mark the curvatures of the last three profiles of Fig. 3.

Conclusions. In this paper, we have given the first analytical description of axisymmetric interface tips, created by a converging viscous flow. The shape of the interface near the tip is universal, independent of flow conditions. The tip curvature always remains finite, but increases dramatically with

flow strength. Thus although surface tension always keeps the interface smooth, the continuum hypothesis will in many cases fail in practice. Moreover, it follows that the numerical solution of free-surface problems places extreme demands on the spatial resolution. Our analytical solution can be used to provide effective boundary conditions on intermediate scales, to drastically reduce the necessary numerical effort. Numerical calculations agree well with our experiments, which resolve the tip size down to a few microns.

Using the theoretical framework established in this paper, we hope to address the important issue of tipstreaming, used to produce colloidal drops (3, 7), and studied numerically in (37, 38). The same is true for a much wider class of flows involving electric fields, where the analogous flow is known as the cone-jet mode (12, 39). In (3), it was shown that in a suitable microfluidic geometry (see Fig. 1, bottom), the transition to tipstreaming is of second order, i.e. the ejected thread can be arbitrarily thin. Thus we expect the transition between a tipstreaming state and a closed tip to be continuous, which means that previous theories of tipstreaming, based on slender-body theory (40, 41), suffer from the same shortcomings as for a tipped state. By adding an inner fluid to our description, we expect to be able to include the thread into our theory, finally being able to address the tipstreaming problem in a consistent fashion.

Materials and Methods

The idea of the boundary integral Eq. (2), is to use the linearity of the Stokes equation to write the velocity as a superposition of an externally imposed velocity field $\mathbf{v}^{(ext)}$, and the velocity $\mathbf{v}^{(J)}$ produced by surface tension. The latter can be seen as coming from a collection of point forces of strength $\gamma\kappa\mathbf{n}$, distributed over the surface S . This makes $\mathbf{v}^{(J)}$ a superposition of *Stokeslets* \mathbf{J} , integrated over the free surface S , with $\gamma\kappa\mathbf{n}$ as a weight.

The kernels in the boundary integral description Eq. (2) are (32)

$$J_{ij}(\mathbf{r}) = \frac{1}{8\pi} \left[\frac{\delta_{ij}}{r} + \frac{r_i r_j}{r^3} \right], \quad K_{ijk}(\mathbf{r}) = -\frac{3}{4\pi} \frac{r_i r_j r_k}{r^5}, \quad [12]$$

where $\mathbf{r} = \mathbf{x}_1 - \mathbf{x}_2$, so that $\mathbf{J} \cdot \mathbf{f}(\mathbf{x}_2)$ is the velocity at \mathbf{x}_1 , generated by a point force \mathbf{f} at \mathbf{x}_2 , and $\mathbf{K} \cdot \mathbf{f}(\mathbf{x}_2)$ is the stress at \mathbf{x}_1 . The factor $1/2$ on the left-hand side of Eq. (2) and the second integral over the stress $\mathbf{K} \cdot \mathbf{n}$ corrects for the jump in viscosity between the two phases.

Putting $\mathbf{x}_1 = (y_1, 0, x_1)$, $\mathbf{x}_2 = (y_2 \cos \theta, y_2 \sin \theta, x_2)$, and $\bar{\mathbf{n}} = (-h'(z), 1)$, we can perform the integration with respect to the azimuthal angle θ :

$$\mathbf{j} = y_2 \int_0^{2\pi} \mathbf{J} \cdot \bar{\mathbf{n}} d\theta_2, \quad [13]$$

and analogously for \mathbf{K} , yielding the kernels \mathbf{j} and \mathbf{k} in Eq. (3) and Eq. (4). They result in well-known expressions (32) involving elliptic integrals in terms of the (logarithmic) coordinates l_1, s_1 and l_2, s_2 .

The entries in Eq. (6) are calculated by performing the integrals over l_2 assuming a conical interface with slope $s_1 = s(l_1)$. This results in integrals over the kernels, such as $\zeta_2 k_{zz}(s_1, \Delta)$ for the case of K_1 , but as a function of $\Delta = l_2 - l_1$ alone. The dominant contribution to the integral comes from a central region of size s_1 . At higher orders, contributions for $\Delta \geq 1$ also need to be taken into account, but can be disregarded here.

To capture contributions at scale s , we put $\xi = \Delta/s$ and expand the kernel:

$$\zeta_2 k_{zz} = K_1^{(-1)}(\xi) s^{-1} + K_1^{(0)}(\xi) + \dots$$

Here $K_1^{(-1)}(\xi)$ is an even function, which is expressible in terms of the elliptic integrals $E\left(2/\sqrt{4+\xi^2}\right)$ and $K\left(2/\sqrt{4+\xi^2}\right)$ (42).

396 Thus the integral becomes

$$397 \int_{-\infty}^{\infty} \zeta_2 k_{zz} d\Delta = \int_{-\infty}^{\infty} K_1^{(-1)} d\xi + O(s^2) = \frac{1}{2} + O(s^2),$$

398 as shown by an explicit calculation. The remaining integrals to
399 calculate $K_2 \dots K_4$ can be evaluated in a similar fashion, expanding
400 the integrand in s at constant ξ .

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