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A Method for Attitude Control Based on a Mathematical Model for an Inverted Pendulum-Type Mobile Robot

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Article Info	ABSTRACT	

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Keyword:

Attitude control Disturbance Inverted pendulum Mathematical model Mobile robot PID controller Amethod for attitude control based on a mathematical model for an inverted pendulum-type mobile robot was proposed. The inverted pendulum-type mobile robot wasdesigned and the mathematical modeling was conducted. The parameters of the mobile robot were estimated and the state-space model of mobile robot was obtained by the substitution of the estimated parameters into the mathematical model. The transfer function of the mobile robot is applied to generate the root-locus diagram used for the estimation of the gains of the PID controller. The attitude control method including a PID controller, non-linear elements, and integral saturation prevention was designed and simulated. The experiment was conducted by applying the method to the mobile robot. In the attitude control experiment, the performance of attitude recovery from $\pm 12^{\circ}$ tilted initial statewith a settling time of 0.98s and a percent overshoot of 40.1% was obtained. Furthermore, the attitude maintaining robustness against disturbance wasverified.

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1. INTRODUCTION

In recent times, several platforms for mobile robot have been proposed by many manufacturing companies. The PUMA, which is a vehicle-type mobile robot for short distance transportation, is developed by Segway. The TBot, developed by IHMC, is an inverted pendulum-type mobilerobot for military applications and is a reconfigurable robot that can switch between a four-wheeled and a two-wheeled configuration. As an industrial service robot, the EMIEW1, having a humanoid-type upper body and a two-wheel-type lower body, and the EMIEW2 having a humanoid-type body are proposed by Hitachi. In recent times, numerous researches has being conducted on inverted pendulum-type two-wheeled mobile robots because of their advantages such as low power consumption, light weight, simple configuration, excellent expandability, and a wide area of applications [1], [2], [3], [4], [5]. Since the inverted pendulum-type mobile robot has unstable characteristics in the system configuration, high technologies such as precision attitude sensing and control algorithm are required for robust attitude balance [6], [7], [8]. Furthermore, a combined design including the mechanism, hardware, and software is required [9].

In this study, an attitude control method based on a mathematical model for an inverted pendulumtype mobile robot is proposed. The inverted pendulum-type mobile robot was designed and parameters were estimated by mathematical modeling. The state-space model for mobile robot was obtained by the substitution of the estimated parameters into the mathematical model. The transfer function of mobile robot is applied to generate the root-locus diagram used for the estimation of the gains of the PID controller. The attitude control method including a PID controller, non-linear elements, and integral saturation prevention

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was designed and simulated. The experiment was conducted by applying the method to the mobile robot. In the attitude control experiment, the performance of attitude recovery from $\pm 12^{\circ}$ tilted initial state and the attitude maintaining robustness of the robot against disturbance was verified.

2. CONFIGURATION OF INVERTED PENDULUM TYPE MOBILE ROBOT

The configuration of an inverted pendulum-type mobile robotis as follows: (Figure 1), a sensing module for the sensing the attitude of the body, a moving module for moving the body and generating the inertia force, a control module for controlling the attitude balance, and a communication module for transferring the data between the mobile robot and the main computer.



Figure 1. Configuration of an inverted pendulum-type mobile robot; (a) control diagram, (b) robot platform

3. MATHEMATICAL MODEL

The mathematical model of the inverted pendulum-type mobile robot is obtained as the state-space model with two degrees of freedom using the free-body and the kinematic diagram as shown in Figure 2.



Figure 2. Free-body and kinematic diagram of inverted pendulum-type mobile robot; (a) wheel, (b) body

From Figure 2(a), the equation for the motion of the wheel is obtained using the horizontal and the rotational components as described in equation (1) [8].

$$2(mr^2 + I_{\omega})\theta = 2T_{\omega} + R_H r \tag{1}$$

Where, m is the mass, r is theradius, I_w is the mass moment of inertia, and θ is the angle of rotation of the wheel, respectively. T_w is the torque generated by the wheel motor and R_H is the sum of the horizontal components of the reaction force between the two wheels and the body. It is assumed that the wheel has only a rotational motion without a sliding motion.

From Figure 2(b), the equation for the motion of the body is obtained using the horizontal and the rotational components as described in equation (2).

$$(I_{p} + ML^{2})\dot{\phi} + MrL\dot{\theta}\cos\phi - MgL\sin\phi = -2T_{\omega}$$
⁽²⁾

Where, I_p is the mass moment of inertia, M is the mass, and φ is the angle of rotation of the body, respectively. L is the distance between the center of mass of the wheels (A) and the center of mass of the body (G).

From Figure 2(a) and 2(b), $R_{\rm H}$ is obtained using the horizontal and rotational motion of the wheel and body as described in equation (3).

$$R_{H} = -Mr\ddot{\theta} - ML\cos\phi\dot{\phi} + ML\sin\phi\dot{\phi}$$
(3)

By substituting the equation (3) in equations (1) and (2), and linearizing at the equilibrium point $\varphi = 0$, the equations of motion for the wheel and the body are simplified as follows:

$$[(2m+M)r^{2}+2I_{\omega})]\theta+MrL\phi=2T_{\omega}$$
(4)

$$(I_p + ML^2)\phi + MrL\theta - MgL\phi = -2T_\omega$$
⁽⁵⁾

In equation (4) and (5), wheel torque (T_w) can be described using the equivalent circuit of the wheel motor as shown in Figure 3.



Figure 3. Equivalent circuit of the DC motor

By ignoring inductance and the coefficient of friction of the motor, applying Kirchhoff's voltage law, and using the equation for the moment at the axis of the motor, the load torque is calculated as follows:

$$T_L = -\frac{K_T K_b}{R_a} \omega_m + \frac{K_T}{R_a} V_a - J_m \theta_m^{'}$$
(6)

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Where, K_T is the torque constant, K_b is the back emf constant, R_a is the resistance, V_a is the applied voltage, and J_m is the moment of inertia of the motor.

When the gear ratio is n, the wheel torque (T_w) and the load torque (T_L) are related by the equation $T_w=nT_L$. The angular displacement of the motor (θ_m) and the angular displacement of the wheel (θ) are related by the equation $\theta = (\theta_m/n) + \varphi$, where the initial angular displacement is φ . By substituting these relations in equation (6), the wheel torque is described by the following equation:

$$T_{\omega} = -\frac{K_T K_b}{R_a} n^2 \dot{\theta} + \frac{K_T K_b}{R_a} n^2 \dot{\phi} + n \frac{K_T}{R_a} V_a - J_m n^2 \dot{\theta} + J_m n^2 \dot{\phi}$$
(7)

By substituting the equation (7) in equations (4) and (5), the equations of motion are completed. The state-space model is obtained by applying the system parameters, which are calculated and measured [7], as described in equation (8).

$$X = AX + Bu,$$

$$Y = CX,$$

$$X = \begin{bmatrix} \theta \cdot \dot{\theta} \phi \dot{\phi} \end{bmatrix}^{T},$$

$$u = V_{a}.$$

(8)

Where we have:

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & -13.41 & 46.63 & 13.41 \\ 0 & 0 & 0 & 1 \\ 0 & 4.45 & 50.82 & -4.45 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 14.14 \\ 0 \\ -4.69 \end{bmatrix}, \text{ and } C = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}.$$

4. CONTROL METHOD AND EXPERIMENT

Although the state-space model has two degrees of freedom, the state-space equation is converted to the transfer function as given in equation (9), and the gains of the PID controller can be estimated using the root-locus diagram generated from this transfer function.

$$G(s) = \frac{\Phi(s)}{V_a(s)} = \frac{4.69 s}{s^3 + 17.86 s^2 - 50.82 s - 889.2}$$
(9)

The mobile robotis unstable because it has the poles on the right plane as shown in Figure 4(a). Using the root-locus diagram, the range of the proportional gain for the stability of the system can be estimated in the different cases where the PI or the PID controlleris used (Figure 4).



Figure 4. Root-locus diagram for the mobile robot according to the type of controller used; (a) P controller, (b) PI controller, (c) PID controller

Root-locus diagram of the mobile robot can be changed by added zeros and poles using the PI or PID controller as shown in Figures 4(b) and 4(c). Equations (10) and (11) describe the transfer function of the PI and PID controller. From these equations, the positions of the zeros can be determined by the gain of the controller.

$$G_{PI}(s) = \frac{K_{p}(s + K_{i}/K_{p})}{s}$$
(10)

$$G_{PID}(s) = \frac{K_d(s^2 + K_p/K_d s + K_i/K_d)}{s}$$
(11)

Where, K_p is the propotional gain, K_i is the integral gain, and K_d is the derivative gain.

A transfer function model cannot reflect the initial state of the tilted body. By applying the statespace model and the feedback from the state variables, the attitude control system including a PID controller, non-linear elements, and integral saturation prevention is constructed in the Simulink as shown in Figure 5.



Figure 5. Attitude control system constructed in the Simulink

In Figure 4, the proportional gain is estimated to be 200 for a percent overshoot of 30%. The integral and derivative gains are determined by the simulation of the root-locus diagram based on the estimated proportional gain. Using the estimated gain parameters, the performance of the attitude control systemis simulated with and without the adoption of the non-linear elements as shown in Figure 6.

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Figure 6. Simulation result of attitude control system with and without the non-linear elements

The experiment is conducted by applying the developed attitude control system having the estimated control gain to the mobile robot. The initial state of the mobile robot is tilted by $\pm 12^{\circ}$ and the angle is reduced and maintained at zero by attitude control. Figure 7 shows the performance of attitude control using the PID gains obtained by simulation including the compensationfor disturbance. The gains of K_p, K_i, and K_d are 1.1, 17, and 0.01 respectively. Table 1 lists the performance of attitude control determined from the experiment and simulation.



Figure 7. Attitude control performance using the PID gainsobtained from simulation

Table 1. Performance of attitude control							
Variable	Rising Time(s)	%OS (%)	Settling Time (s)	Steady State Error (deg)			
Experiment	0.14	40.1	0.98	±0.3			
Simulation	0.13	37.0	0.83	± 0.1			

5. CONCLUSION

The attitude control method based on a mathematical model for an inverted pendulum-type mobile robot was proposed. The inverted pendulum-type mobile robot was designed and the mathematical modeling was conducted. The parameters of mobile robot were estimated and the state-space model of mobile robot was obtained by the substitution of the estimated parameters into the mathematical model. The transfer function of mobile robot is applied to generate the root-locus diagram used for the estimation of the gains of the PID controller. The attitude control method including a PID controller, non-linear elements, and integral saturation prevention was designed and simulated. The experiment was conducted by applying the system to

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