A model for calculating the mechanical demands of overground running

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1 Abstract

2 An energy-based approach to quantifying the mechanical demands of overground, constant 3 velocity and/or intermittent running patterns is presented. Total mechanical work done (W_{total}) is 4 determined from the sum of the four sub components: work done to accelerate the centre of mass 5 horizontally (W_{hor}) , vertically (W_{vert}) , to overcome air resistance (W_{air}) and to swing the limbs 6 (W_{limbs}) . These components are determined from established relationships between running 7 velocity and running kinematics; and the application of work-energy theorem. The model was applied to constant velocity running $(2-9 \text{ m} \cdot \text{s}^{-1})$, a hard acceleration event and a hard deceleration 8 9 event. The estimated W_{total} and each sub component were presented as mechanical demand (work 10 per unit distance) and power (work per unit time), for each running pattern. The analyses 11 demonstrate the model is able to produce estimates that: 1) are principally determined by the 12 absolute running velocity and/or acceleration; and 2) can be attributed to different mechanical demands given the nature of the running bout. Notably, the proposed model is responsive to varied 13 14 running patterns, producing data that are consistent with established human locomotion theory; 15 demonstrating sound construct validity. Notwithstanding several assumptions, the model may be applied to quantify overground running demands on flat surfaces. 16

17

18 Keywords

- 19 energetics, power, external load, locomotion, match analysis
- 20 21

22 Introduction

23 Quantifying the loads athletes experience during training, competition and/or in research settings 24 is routine practice, with several methods employed across settings (Lambert & Borresen, 2010). 25 The value, utility, practicality, limitations and future directions of load quantification methods 26 have been topics of discussion for several years (Aughey, 2011; Bourdon et al., 2017; Cummins, 27 Orr, O'Connor & West, 2013; Gray, Shorter, Cummins, Murphy & Waldron, 2018; Lambert & 28 Borresen, 2010). Where training theory is considered a simple 'dose-response' relationship, there is consensus that the exercise 'dose' experienced during training or competition can be described 29 30 in two ways; through objective measures of the work performed by the athlete (external load) or 31 as the relative biological (both physiological and psychological) stressors imposed on the athlete 32 (internal load) (Bourdon et al., 2017). The 'response' may be described by changes in performance 33 and/or adaptation, which notably, can be positive (e.g. performance increase, favourable 34 physiological adaptation, readiness to train) or negative (e.g. symptoms of fatigue, overuse injury, 35 reduced performance). Consistent with this understanding, several studies have implicated training 36 load as having influence over performance outcomes (Jobson, Passfield, Atkinson, Barton & Scarf, 2009; Taha & Thomas, 2003), athlete wellbeing (Lathlean, Gastin, Newstead & Finch, 2019), 37 38 fatigue/readiness to perform (Halson, 2014) and injury (Schwellnus et al., 2016; Soligard et al., 39 2016). To gain such insights, simultaneous monitoring of both external and internal load is 40 recommended, as this permits the evaluation of psychophysiological stress relative to the work 41 done. Indeed, reduced homeostatic disturbance to a given absolute work rate is a hallmark response 42 to exercise training (Blomqvist & Saltin, 1983; Holloszy & Coyle, 1984). This speaks to the importance of adopting valid and reliable load monitoring methods (Lambert & Borresen, 2010). 43

45 The introduction of micro-technology devices (small units co-housing a global positioning system 46 (GPS) receiver and various micro-electrical mechanical systems) designed for sporting 47 applications has attracted considerable interest and discussion on how such data can and should be 48 treated to understand performance, guide training design and inform player management decisions, 49 particularly in field based team sports (Aughey, 2011; Cummins et al., 2013), where traditional 50 load monitoring methods e.g. heart rate monitoring, are unsuitable given the intermittent nature of 51 these sports (Bangsbo, Mohr & Krustrup, 2006). Notwithstanding the limitations of micro-52 technology devices (Malone, Lovell, Varley & Coutts, 2017), it would seem they continue to be 53 used across many team sports as they readily provide kinematic summaries (time, distance, 54 velocity, acceleration) of the gross locomotor patterns during field-based training and competition. Despite some microtechnology derived metrics demonstrating relationships with measures of 55 56 acute internal load (Impellizzeri, Rampinini, Coutts, Sassi & Marcora, 2004) and/or readiness to 57 perform (Young, Hepner & Robbins, 2012), the literature highlights several shortcomings and opportunities to improve common techniques (Bourdon et al., 2017; Furlan, Osgnach, Andrews & 58 59 Gray, 2014; Gray et al., 2018). For example, Bourdon et al. (2017) identify that the manner in 60 which commercial systems determine and report sprint and/or acceleration efforts, is often at odds 61 with how a coaches view said efforts, leading to misinterpretation. Similarly, Gray et al. (2018) 62 describe how the use of speed/acceleration zones (i.e. sample by sample binning of data according to speed or acceleration) fragment work bouts, rather than painting clear pictures of the work 63 64 performed. Based on these discussions, the future of external load monitoring in team sports appears to destined for improved wearable sensors (with technological advancements) and 65 66 advanced modelling techniques applied to present meaningful summary data to coaches and 67 athletes (Bourdon et al., 2017).

69 In cycling, ergometers and power meters provide measures of mechanical work (total external 70 load) and power-time curves that are readily analysed to describe the intensity and distribution of 71 work. Whilst these technologies do not capture internal power (Brooks, Andrews, Gray & 72 Osborne, 2013), it is arguably the gold standard method of measuring external load for cycling 73 exercise. Intuitively, the work/power method summates rather than fragments data, and uses 74 dimensionally appropriate units (as opposed to arbitrary units) for external load quantification. 75 Measuring mechanical work and power during overground running is not nearly as simple, but is 76 possible. Valuable insights such as the costly nature (both mechanically and metabolically) of 77 accelerated and decelerated running (Osgnach, Poser, Bernardini, Rinaldo & di Prampero, 2010; 78 Pavei et al., 2019; Zamparo et al., 2019) have resulted from energy-based analyses, as such, 79 pursuing a field-based method of quantifying external load in terms of work and power seems 80 advantageous from multiple perspectives. Gray et al. (2018) proposed that following sport-specific 81 temporal classification of data sets into discrete movement categories e.g. walking, running, 82 colliding, wrestling; a model specific to each movement category could be applied to provide a 83 work-energy based description of each bout. Subsequent summation of all occurrences would yield the total 'load' of the bout. 84

85

The movement demands of field-based team sports are well documented (Bangsbo et al., 2006; Duthie, Pyne & Hooper, 2003; Gabbett, King & Jenkins, 2008; Wisbey, Montgomery, Pyne & Rattray, 2010), with many match analysis studies identifying that a large proportion of play is spent in low-intensity locomotor activities (walking and jogging or $< 3.5 \text{ m} \cdot \text{s}^{-1}$) interspersed with brief (~3- 10 s) repeated bouts of high-intensity locomotor efforts (high speed running, explosive

91 acceleration/deceleration). Given that the forward running gait is the predominant 'purposeful 92 movement' in team sport match play (Bloomfield, Polman & O'Donoghue, 2007), a work-energy 93 model for this specific movement category is likely to be an essential component of the external 94 load profile of most field sports. Based on the Konig Theorem, Gray et al. (2018) conceptualised 95 a model for the determination of mechanical work done during overground forward running. This 96 study aims to apply this model to GPS derived velocity-time data to describe the mechanical power 97 and mechanical demand during three conditions: 1) constant velocity running (simulated); 2) a maximal acceleration (simulated 40 m sprint); and 3) an intense deceleration (during on field 98 99 training). This analysis serves to demonstrate how an energy-based approach can quantify the 100 external load during over ground running of varied nature. It is hypothesised that the model will 101 produce estimates of mechanical power for continuous and intermittent running bouts, that are 102 appropriate for load monitoring applications.

103 Methods

104 Theory

The total mechanical work (W_{total}) done during running can be partitioned into external work (W_{ext}) and internal work (W_{int}) (Pavei et al., 2019a; Saibene & Minetti, 2003), where W_{ext} is the work done to accelerate the centre of mass (COM) with respect to the environment and W_{int} is the work associated with the acceleration of body segments with respect to the COM. Therefore, total mechanical work is given by:

110
$$W_{total} = W_{int} + W_{ext}$$
(1)

Furthermore, work done can be defined as either positive or negative. Positive work (W^+) is done when the kinetic (*KE*) and/or potential energies (*PE*) of a mass are increased. Conversely, negative work (W⁻) is done when the kinetic (*KE*) and/or potential energies (*PE*) of a mass are decreased.
These principles underpin all subsequent discussion.

115

In overground running on a level surface the COM is accelerated in the horizontal and vertical planes (Cavagna, Saibene & Margaria, 1964). Additionally, even in the absence of wind, air poses a resistive force to the motion of the COM (di Prampero, 1986). Therefore, W_{ext} can be considered a function of the work done on the COM in the horizontal plane (W_{hor}), vertical plane (W_{vert}) and to overcome air resistance (W_{air}). Therefore, external work is given by:

121
$$W_{ext} = W_{hor} + W_{vert} + W_{air}$$
(2)

122 Internal work (W_{int}) is typically determined from changes in segment energies derived from motion 123 analysis (Pavei et al., 2019; Zamparo et al., 2019). However, Minetti (1998) provides a model 124 equation to predict W_{int} from velocity, stride frequency, duty factor (the percentage of the stride 125 cycle in which a single limb is in the support phase) and a constant reflecting the inertial properties 126 of the limbs. In the absence of uneven terrain, varying loads or changes in wind direction and 127 speed, body mechanics are tightly coupled with forward velocity in running (Gray, Price, & 128 Jenkins, In Press; Lee & Farley, 1998; Mann & Hagy, 1980; Nilsson, Thorstensson & Halbertsma, 129 1985; Saito, Kobayashi, Myashita & Hoshikawa, 1974; Zatsiorsky, Werner & Kaimin, 1994). As 130 such, stride frequency and duty factor are readily modelled from running velocity (Gray et al., In 131 Press), enabling the subsequent determination of W_{int} (Minetti, 1998). As W_{int} is primarily 132 determined by limb kinematics, W_{limbs} is used in the present study to denote this partition.

134 Model Calculations

The velocity-time curve used in the modelling process is assumed to represent the horizontal motion of the COM during forward, overground running on a hard (not able to be deformed), horizontal surface orthogonal to the earth's gravitational field; the runner's sagittal plane (the plane upon which the runner's limbs tend to have their greatest angular motion) assumes a fixed vertical orientation i.e. perpendicular to the running surface.

140

The following sections describe a method of determining W_{total} during an overground running bout, from the velocity-time curve of a GPS receiver. Common to all systems will be a finite sampling frequency, as such the velocity-time curve of any running bout to be analysed will include a finite number of samples (*n*), a fixed time interval (t_i) between samples. The formulae presented herein are written for the *j*th sample, over a period of n, GPS samples.

- 147 Determination of mechanical work and power from GPS velocity data according to the above148 theoretical framework was completed in four steps:
- 149 1. Predicting COM and limb kinematics from GPS velocity
- 150 2. Determining external work from GPS Velocity
- 151 3. Determining internal work from GPS Velocity
- 152 4. Summation to determine total mechanical work and power
- 153
- 154 1. Predicting COM & Limb Kinematics from GPS Velocity
- 155 The kinematics of the COM and the limbs are tightly coupled to running speed. The motion of the
- 156 COM in running is likened to a bouncing ball, where it is lowest during mid support and highest

157 in mid-flight (Farley & Ferris, 1998). Therefore, with each step (half stride) there is a vertical 158 oscillation of the COM, the vertical displacement (Δh , from lowest to highest point) of which, has 159 been shown to vary linearly with movement velocity ($r^2 = 0.444$, p = 0.034, n = 90) (Ito, Komi, 160 Sjodin, Bosco & Karlsson, 1983; Lee & Farley, 1998) according to:

161
$$Dh = -$$

$$Dh = -0.008 + 0.004 \cdot v \tag{3}$$

where Δh is in m, and v in m s⁻¹. 162

163

164 Similarly, temporal limb kinematics have been shown to vary with 'steady state' running velocity. 165 Support duration decreases whilst swing duration is maintained or only modestly decreased at high 166 speeds (Nilsson et al., 1985). The percentage of the stride cycle in which a single limb is in the 167 support phase is termed the duty factor. Consequently, with increasing 'steady state' running 168 velocity, stride frequency (f) increases whilst duty factor (d) decreases. Given f and d are notable 169 determinants of mechanical power in locomotion (Minetti, 1998; Nardello, Ardigo & Minetti, 170 2011), Gray et al. (In Press) have previously established regression equations relating stride 171 frequency and duty factor to running velocity in a sample of male football (soccer) players. The 172 regression equations determined were:

$$f = 0.026 \times v^2 - 0.111 \times v + 1.398 \tag{4}$$

174

$$d = 0.004 \times v^2 - 0.061 \times v + 0.50 \tag{5}$$

where f is in Hz, d is % (in decimal form), and v in $m \cdot s^{-1}$. The application of equations 3, 4 and 5 175 176 will soon be explained.

177

178 2. Determining External Work from GPS Velocity

External work done can be determined from changes in the kinetic (*KE*) and potential energy (*PE*) of the COM (Cavagna et al., 1964). The *KE* of the COM is the vectorial sum of its horizontal (*KE*_{hor}) and vertical (*KE*_{vert}) components, thus W_{hor} is given by the change in *KE*_{hor}. The horizontal velocity of the COM may be approximated by velocity-time data from a micro-technology device. The resolution and sampling frequency of this technology is unlikely to detect within stride fluctuations in COM motion, therefore this data can only be assumed to represent the gross forward velocity, which is important nonetheless. On this basis, *W*_{hor} can be expressed as:

186
$$W_{hor}^{j} = \mathop{a}\limits^{n}_{j=1} 0.5 \left(v_{j+1}^{2} - v_{j-1}^{2} \right)$$
(6)

187 Importantly, where $v_{j+1} > v_{j-1}$ (as for acceleration), positive horizontal work (W_{hor}^+) is done by the 188 body. Where $v_{j+1} < v_{j-1}$ (as for deceleration), negative horizontal work (W_{hor}^-) is done by the body. 189 Furthermore, when determining work done from changes in KE, mass is a scaling factor and has 190 therefore been excluded such that units are J·kg⁻¹.

191

192 With each step taken, the COM rises and falls by a height, Δh (Lee & Farley, 1998). The vertical 193 oscillation of the COM suggests the KE_{vert} and PE of the COM are in continual flux. Additionally, 194 the first law of thermodynamics implies $\Delta PE = \Delta KE_{vert}$, therefore either may be used to 195 approximate W_{vert} . In this approach, ΔPE will be used given Δh can be predicted from velocity 196 using equation 3. ΔPE of the COM from its lowest to highest position and vice versa, equate to 197 the positive vertical work (W_{vert}^+) and negative vertical work (W_{vert}) done by the body, respectively. Assuming, the COM rises and falls the same height in a step, it holds that $|W_{vert}^+|$ 198 = $|W_{vert}|$. Thus, either can be expressed as: 199

200
$$\left| W_{vert^{+}}^{j} \right| = \left| W_{vert^{-}}^{j} \right| = \sum_{j=1}^{n} \left(2 \cdot g \cdot \mathsf{D}h_{j} \cdot f_{j} \right)$$
(7)

where Δh_j and f_j are predicted from v_j using equations 3 and 4, respectively. Similar to equation 6, when determining work done from changes in PE, mass is a scaling factor and has again been excluded such that units are J·kg⁻¹.

204

Air resistance (F_{air}) is an external force applied by the volume of air that meets and passes around the surface of a body. It can be mathematically expressed as a function of ambient air density (ρ) , projected frontal surface area (A_p) , the square of the relative air speed (S) and a drag coefficient (C_d) according to:

$$F_{air} = 0.5 \times \Gamma \times A_{\Gamma} \times S^2 \times C_d \tag{8}$$

210 Air density varies with *T* and *BP*, therefore with knowledge of these values, ambient air density 211 (ρ) can be estimated according to:

212
$$\Gamma = \frac{273 \times \Gamma_o \times BP}{760 \times T}$$
(9)

213 with the unit kg·m⁻³, where *BP* is in mmHg, *T* is in °C and $\rho_o = 1.293$ kg·m⁻³ (air density at sea 214 level and 273 K).

215

Projected frontal surface area of a human running is ~26 % of total body surface area (*BSA*) (Davies, 1980; Pugh, 1971, 1976; Shanebrook & Jaszczak, 1976), which can be determined using established prediction equations (DuBois & DuBois, 1916; Shuter & Aslani, 2000). Applying a BSA prediction equation (Shuter & Aslani, 2000), A_p can be determined according to:

220
$$A_{r} = 0.26 \left(94.9 \times ht^{0.655} \times M^{0.441} \right)$$
(10)

221 with the unit m², where ht = standing height in m and M = body mass in kg.

Using varied methodological approaches, the C_d for humans running ranges from 0.7 to 1.1 (Davies, 1980; Pugh, 1971; Shanebrook & Jaszczak, 1976; Walpert & Kyle, 1989). In the present model, $C_d = 1$ will be adopted.

226

In calm air, the movement velocity (*v*) of a runner determines the relative air speed, thus v = S (di Prampero, 1986). Under these conditions, the mechanical work done to overcome air resistance (*W_{air}*) is proportional to the cube of the runners forward velocity i.e. v^3 and can be expressed as: 230

231
$$W_{air}^{j} = \sum_{j=1}^{n} \left(\frac{0.5 \cdot \Gamma \cdot A_{r} \cdot v_{j}^{3} \cdot C_{d_{j}} \cdot t_{i}}{M} \right)$$
(11)

232 with the unit J·kg⁻¹, where ρ , A_{ρ}, v, C_d, t_i and M are substituted as previously defined.

233

234 3. Determining Internal Work from GPS Velocity

Internal work primarily describes the work done to swing the limbs (W_{limbs}) and is typically determined from changes in segment energies derived from motion analysis. However, Minetti (1998) provides a model equation to predict the mechanical work done to swing the limbs, per unit distance travelled (D_{limbs}), in walking and running from velocity, stride frequency and duty factor as follows:

240
$$D_{limbs} = q \cdot v^2 \cdot f\left(I + \left(\frac{d}{I - d}\right)^2\right)$$
(12)

where q = 0.1, and is a constant reflecting the inertial properties of the limbs and the mass partitioning between the limbs and the rest of the body (Minetti, 1998) (units are J·kg⁻¹·m⁻¹). This equation allows within and between segment energy transfer and takes the absolute sum of positive and negative work performed by the limbs (Minetti, 1998; Nardello et al., 2011). On this understanding, W_{limbs} can be expressed as:

246
$$W_{limbs}^{j} = \sum_{j=1}^{n} \left(q \cdot v_{j}^{3} \cdot f_{j} \left(I + \left(\frac{d_{j}}{I - d_{j}} \right)^{2} \right) \cdot t_{i} \right)$$
(13)

247 where f_j and d_j are predicted from v_j using equations 4 and 5, respectively (units are J·kg⁻¹).

249 4. Summation to Determine Total Mechanical Work, Power and Demand

Equations 6, 7, 11 and 13 define components $(W_{hor}^+, W_{hor}^-, W_{vert}^+, W_{vert}^-, W_{air}$ and $W_{limbs})$ of the total mechanical work done (W_{total}) for running at a given velocity. As such, W_{total} can be expressed as:

253
$$W_{total}^{j} = \mathop{\stackrel{n}{\Leftrightarrow}}_{j=1}^{n} \left(\left| W_{vert^{+}}^{j} \right| + \left| W_{vert^{-}}^{j} \right| + \left| W_{horiz}^{j} \right| + W_{limbs}^{j} + W_{air}^{j} \right)$$
(14)

254 where W_{total} is in J·kg⁻¹.

255

The total mechanical power (P_{total}) can be determined by dividing by the time interval according to:

258
$$P_{total}^{j} = \frac{W_{total}^{j}}{t_{i}}$$
(15)

units are W·kg⁻¹. To determine the mechanical power of any sub component in the model e.g. P_{hor}^+ from W_{hor}^+ , the same approach can be applied.

261

262 The total mechanical demand (D_{total}) can be determined by dividing mechanical power (P_{total}) by 263 the running velocity according to:

$$264 D_{total}^{j} = \frac{P_{total}^{j}}{v^{j}} (16)$$

units are $J \cdot kg^{-1} \cdot m^{-1}$, a customary unit for the mechanical and metabolic cost of locomotion (Minetti, 1998). To determine the mechanical demand of any sub component in the model e.g. D_{hor}^{+} from P_{hor}^{+} , the same approach can be applied.

268 Participants

For condition 1), data that simulated constant velocity running were manually developed therefore no participants were required. For conditions 2) and 3), ten elite Australian football players were recruited from an Australian Football League (AFL) club to participate. The participants represented a cross section of age, size, and running ability of elite Australian football players (mean \pm SD age: 25.4 \pm 4.1 years, body mass: 89.3 \pm 11.4 kg, stature: 188.9 \pm 7.1 cm). Informed consent was gained prior to participation and the study was approved by an ethics committee of The University of Queensland.

276 Procedures

Data sets simulating constant velocity running at 2, 3, 4, 5, 6, 7, 8, 9 and 10 m·s⁻¹ were prepared for analysis in R (R, Vienna, Austria), which determined mechanical work done based on the model described above. Environmental conditions were standardised (BP= 760 mmHg, T= 23° C and no wind) and mean stature (189 cm) and body mass (89.3 kg) of the participants were used in the calculations. The relationships between constant velocity running and mechanical power are presented.

283

GPS data (SPIpro, GPSports, Canberra, Australia) collected at 5 Hz during a pre-season sprint
testing session (3 x 40 m sprints on an outdoor tartan athletics track) were downloaded (GPSports,

Team AMS, Canberra, Australia) and reviewed to set parameters for an exponential function(Chelly & Denis, 2001; P. E. di Prampero et al., 2005) that represented the group's sprint

288 performance. This was:
$$v_t = v_{max} \times \left(I - e^{\frac{-t}{t}} \right)$$
 (17)

where v_t is the modelled running velocity in m·s⁻¹, v_{max} is the maximal velocity reached during the sprint in m·s⁻¹, and τ is the time constant in s. The mean \pm SD v_{max} of the participant group was 9.16 \pm 0.42 m·s⁻¹, which was substituted into equation 17, along with $\tau = 1.4$. The modelled velocity-time curve (reproduced at 5 Hz) was visually inspected and considered to adequately represent the sprint performance of the participant group (Figure 1). This velocity-time data was then imported for analysis in R, as described above. The modelled changes in mechanical work and power over the duration of the simulated sprint are presented.

297 GPS data recorded during a regular season, field-based training session were downloaded and 298 reviewed to identify each participant's peak deceleration not attributed to a collision or fall. This discrete deceleration event was exported to Microsoft Excel (Microsoft Corp., Redmond, USA) 299 300 where kinematic variables used to describe the nature of deceleration events were determined; duration (s), initial velocity $(m \cdot s^{-1})$, final velocity $(m \cdot s^{-1})$ and peak deceleration $(m \cdot s^{-2})$. These 301 302 events were then opened for analysis in R. The application used the raw, exported 5 Hz velocity-303 time curves to determine the mechanical work done based on the model described above. 304 Participant characteristics (stature, body mass and maximum running velocity) were individualised 305 in this deceleration analysis. The modelled changes in mechanical work and power during the 306 participant's decelerations are presented. The data of Participant 6 are presented graphically to illustrate how the model operates. Participant 6 was selected on the basis of mass, stature and sprint 307

ability, which are consistent with mean values for elite Australian football players (Buttifant, 1999;
Young et al., 2005).

310

311 **Results**

Consistent with the units defined in equations 14, 15 and 16, all presented estimates of

313 mechanical work, power and demand are expressed relative to body mass for comparative

314 purposes.

315 Constant Velocity Running

During simulated constant velocity running W_{hor}^+ and W_{hor}^- , are equal to zero. Figure 2 shows the 316 317 changes in D_{vert}^+ , D_{vert}^- , D_{air} and D_{limbs} (components of mechanical demand) for constant velocity running from 2 - 10 m·s⁻¹. D_{total} was minimised at ~4 m·s⁻¹ before increasing curvilinearly with 318 319 running velocity (Figure 3). P_{total} (total mechanical power) increased in an exponential manner from ~4.4 W·kg⁻¹ at 2 m·s⁻¹, up to ~42 W·kg⁻¹ at 10 m·s⁻¹. At a low running speed of 3 m·s⁻¹, the 320 mechanical work done to raise and lower the COM $(W_{vert}^+ \& W_{vert})$ accounted for ~68% of the 321 322 total mechanical work done, followed by W_{limbs} and W_{air} , with ~30% and 2% respectively. At 9 m·s⁻¹, W_{limbs} was the primary contributor to mechanical demand (~77%), followed by W_{vert}^+ & 323 324 W_{vert} (~15%) and W_{air} (~8%). The relative contributions from each component in the model for 325 running velocities between 2 and 10 m \cdot s⁻¹ are shown in Figure 4.

326

**** Figure 2, 3 & 4 near here***

327 Acceleration

328 Mechanical demand reached a peak of 6.8 J·kg⁻¹·m⁻¹ just 0.4 s into the maximal 40 m sprint (~6 s 329 in total), at a horizontal velocity of 2.3 m·s⁻¹ and an acceleration of 9.87 m·s⁻², before reducing to

almost half of this value (3.45 J·kg⁻¹·m⁻¹) as v_{max} was attained (Figure 5b). Mechanical power 330 increased rapidly over the first second, followed by a slow progression toward a peak value of ~31 331 $W \cdot kg^{-1}$ at a horizontal velocity and acceleration of ~9 m $\cdot s^{-1}$ and 0.24 m $\cdot s^{-2}$, respectively (Figure 332 5c). The total work done over the whole sprint was estimated to be 160.6 J \cdot kg⁻¹. Of this, 54.2% 333 334 was attributed to swinging the limbs back and forth (W_{limbs}), 25.1% to accelerate the COM horizontally (W_{hor}^+) , 14.7% to accelerate and decelerate $(W_{vert}^+ \& W_{vert})$ the COM vertically and 335 336 5.8% to overcome air resistance (W_{air}). Figure 7a shows the mechanical power curves for each 337 component of the model during the simulated sprint.

338 ****Figure 5 near here****

339 Deceleration

The mean \pm SD duration (s), initial velocity (m·s⁻¹), final velocity (m·s⁻¹) and peak deceleration 340 341 $(m \cdot s^{-2})$ of the deceleration curves collected from the team training session were 2.1 ± 0.2 s, 6.4 ± 1.1 m·s⁻¹, 1.2 \pm 0.8 m·s⁻¹ and -5.3 \pm -0.6 m·s⁻², respectively. Figure 6a shows the velocity-time 342 curve of Participant 6 during hard voluntary deceleration. All other participants had similar shaped 343 344 curves despite some variation in the initial and final velocities. The mechanical demand reached a peak of 8.1 J·kg⁻¹·m⁻¹ just 1.5 s into the 2.4 s deceleration event, at a horizontal velocity of 4.0 345 $m \cdot s^{-1}$ (Figure 6b). This occurred at the same time as the peak deceleration (-6.6 $m \cdot s^{-2}$). Mechanical 346 347 power typically began relatively high (dependent on the initial velocity), increased to a peak under 348 intense deceleration and reduced to a minimum once velocity tended towards a constant, low value. For Participant 6, mechanical power was initially high, but stable at $\sim 22 \text{ W} \cdot \text{kg}^{-1}$, before peaking 349 350 at 43.5 W·kg⁻¹, then returning to zero (Figure 6c). This peak occurred just prior to the peak 351 deceleration. Moreover, for Participant 6, the total work done over the whole 2.4 s deceleration was estimated to be 58 J·kg⁻¹. Of this, ~52% was attributed to decelerating the COM horizontally 352

353 (W_{hor}) , ~32% to swinging the limbs back and forth (W_{limbs}) , ~14% to accelerate and decelerate 354 $(W_{vert}^+ \& W_{vert})$ the COM vertically and 1% to overcome air resistance (W_{air}) . Figure 7b shows the 355 mechanical power curves for each component of the model during Participant 6's deceleration 356 event.

****Figure 7 near here****

357 ****Figure 6 near here****

358

C

359 **Discussion and Implications**

360 This study describes and applies a new energetic approach to model the demands of non-steady state overground running from GPS data, that offers insights into the mechanical demands of 361 362 running. Application of the model to constant velocity, accelerated and decelerated running has 363 demonstrated the manner by which the model quantifies the mechanical demands of varied running 364 patterns. Specifically, the analysis highlights that the model is able to produce estimates of 365 mechanical demand that: 1) are principally determined by the absolute running velocity and/or 366 acceleration; and 2) can be attributed to different mechanical loads on the runner given the nature 367 of the running bout.

368

There is a tenfold variation (1.81- 18.3 W·kg⁻¹) in estimates of total mechanical power for running at 3.6-3.9 m·s⁻¹; largely attributable to whether within and between-segment energy transfer is permitted in the model (Arampatzis, Knicker, Metzler & Bruggemann, 2000). By allowing within and between segment energy transfer when deriving W_{limbs} and taking the absolute sum of positive and negative work throughout, the present analysis yields a mechanical power of ~6 W·kg⁻¹ for running at 3.75 m·s⁻¹. This approach was adopted to permit derivation of metabolic power in future analyses (Zatsiorsky, 1997). Despite the values in this analysis falling neatly within those reported

376 in the literature, the general lack of consensus regarding methodological approach (Arampatzis et 377 al., 2000), makes it difficult to comment on the validity of the mechanical power estimates 378 produced. Nonetheless, applying the model to constant velocity running clearly showed that the 379 mechanical demands of running increased with velocity, independent of acceleration (Figure 3). 380 As W_{int} is intuitively related to stride frequency, it is not surprising that W_{int} tends to increase with 381 speed for both walking and running (Nardello et al., 2011). In contrast, W_{ext} tends to decrease with 382 constant velocity running (Cavagna & Kaneko, 1977). The greater increases in W_{int} compared to 383 W_{ext} result in overall increases in W_{total} with velocity. Figure 4 reflects these well-accepted concepts 384 in the human locomotion literature, with P_{total} primarily attributed to P_{vert} at low running velocities 385 and P_{limbs} at high running velocities.

386

387 Collectively, the model suggests continual shifts in the primary mechanical demands of the energy 388 expended during intermittent running. The model describes accelerating the COM vertically as the 389 greatest mechanical demand during low velocity, low acceleration running efforts (Figure 4); 390 swinging the limbs as the greatest mechanical demand during high velocity, low acceleration 391 running efforts (Figure 7); and accelerating/decelerating the COM horizontally as the greatest 392 mechanical demand during low-moderate velocity, high acceleration/deceleration running efforts 393 (Figure 7). These general outcomes of the model are consistent with our understanding of human 394 locomotion (Cavagna & Kaneko, 1977; Doke, Donelan & Kuo, 2005; Farley & Ferris, 1998) and 395 the findings of recent experimental work on the sprint acceleration (Pavei et al., 2019) and shuttle 396 running (Zamparo et al., 2019) mechanics/energetics. Indeed, a mechanical power analysis of 397 maximal 20 m sprints using a 35-camera motion capture system reports peak power values of ~30 $W \cdot kg^{-1}$, with the forward (horizontal) acceleration of the COM, vertical acceleration of the COM 398

399 and acceleration of the limbs relative to the COM, accounting for 50%, 9% and 41% of the total 400 power, respectively. To enable comparison, by removing the W_{air} component from the present 401 model and applying it to the velocity-time curve produced by equation 17 over a 3-second period 402 (to simulate a 20 m sprint), W_{hor}, W_{vert} and W_{limbs} were found to account for 49%, 16% and 35%, 403 respectively. Thus, the present model provides field-based estimates of mechanical power 404 partitions in similar proportions to gold standard laboratory measurements. Similarly, the 405 acceleration/deceleration data presented are consistent with the findings of Zamparo et al. (2019); 406 which demonstrates athletic males produce greater mechanical power during maximal deceleration 407 than maximal acceleration.

408

409 It is now commonly accepted that acceleration and deceleration are energetically costly running 410 patterns (Polglaze & Hoppe, 2019). The model estimates D_{total} during constant velocity running at 9 m·s⁻¹ (approximate peak running velocity of elite field sport athletes) to be 3.3 J·kg⁻¹·m⁻¹ (Figure 411 412 3). Notably, this falls short of the D_{total} values observed during maximal accelerations (6.8 J·kg⁻ 1 ·m⁻¹) and decelerations (8.1 J·kg⁻¹·m⁻¹). Moreover, Figures 5 and 6 clearly demonstrate the 413 414 mechanical demand reaches a peak when the rate of change in velocity is greatest. Figure 7 415 confirms it is indeed the W_{hor}^+ and W_{hor}^- components of the model that are responsible for raising 416 the mechanical demand of such running events. These comparisons highlight the model readily 417 captures the 'costly' nature of acceleration and deceleration events. In contrast, the model suggests 418 that in calm conditions overcoming air resistance presents a very minor contribution to the overall mechanical demand of running. Indeed, despite increasing with running velocity, at 10 m·s⁻¹, D_{air} 419 420 accounts for less than 10% of D_{total} (Figure 4), which is also consistent with previous research (di 421 Prampero, 1986; Pugh, 1971; Ward Smith, 1984).

422 Limitations

423 The model proposed herein and its applications are based on the following assumptions:

424 1) The vertical displacement of the COM, stride frequency and duty factor are predicted from 425 forward velocity according to equations 3, 4 and 5. Firstly, these relationships have been derived 426 from constant velocity overground running in sub-elite athletes (Gray, et al., In Press; Lee & 427 Farley, 1998). Pavei et al. (2019) report stride frequency and duty factor during maximal 20 m 428 sprints in a laboratory setting, showing stride frequency is almost constant at ~2 Hz throughout the 429 accelerated running bout; whilst duty factor quickly declined from ~ 0.38 to plateau at ~ 0.2 after 430 ~ 10 m. Applying these values to the first 3 seconds of the 40 m sprint data in this study (to evaluate 431 the error introduced by applying constant speed kinematics to accelerated running) resulted in a 432 mean change in P_{total} of 1.3%, however this was the net effect of up to ~8% underestimation in the 433 initial stages of the sprint and up to $\sim 10\%$ overestimation in the latter stages. To the authors 434 knowledge, no data exists that allows for similar comparisons during deceleration and/or change 435 of direction, as such the magnitude of error introduced for these running patterns is unknown. 436 Secondly, effects of fatigue (Brueckner et al., 1991), size (Saibene & Minetti, 2003), running 437 surface (Lejune, Willems & Heglund, 1998), running ability (Paradisis et al., 2019) and other 438 contextual factors on these kinematic variables are not taken into consideration. With 439 improvements in wearable technology, direct measurement of these variables may replace these 440 prediction equations, however until such time, this serves as a first approximation.

441

Vertical work done by the COM is determined, on the understanding that the COM rises and
falls to the same height in a step. Studies suggest this is a simplification of the 'true' trajectory of
the COM during running (Cavagna, 2006; Ito et al., 1983; Lee & Farley, 1998). Furthermore, the

445 model assumes the runner's sagittal plane is always vertical, such that the oscillation of the COM 446 can be quantified by changes in PE. This assumption, does not consider the observation that 447 runner's lean (change the orientation of the sagittal plane) during 'bend running' and markedly 448 lower their COM during more abrupt changes of running direction. Movement in the coronal plane 449 is assumed to be negligible and given that GPS receivers have insufficient resolution to detect 450 within-stride fluctuations in forward velocity, the positive and negative work associated with the 451 propulsive and braking forces during stance are also negated. These assumptions appear to result 452 in overestimations, based on comparisons with recent experimental works (Pavei et al., 2019), 453 however it is not possible to quantify the magnitude of error this introduces based on current 454 literature.

455

456 3) Mechanical internal work was predicted using the prediction equation of Minetti (1998), which 457 is based on several assumptions itself, namely the four limbs are straight segments with constant 458 inertial properties at all running speeds. This is clearly a simplification of the 'true' limb structure 459 and human gait and it may have led to an overestimation of the mechanical demand of swinging 460 the limbs. The equation has proven a robust alternative to direct measurement during constant 461 speed (Nardello et al., 2011) and short sprint running (Pavei et al., 2019). However, during 462 accelerated running where limb configurations are changing on a step-by-step basis (Nagahara, 463 Matsubayashi, Matsuo & Zushi, 2014; Pavei et al., 2019), the compound factor 'q' decays 464 exponentially from ~0.22 to reach an asymptote of ~0.1 (as in constant speed running). Where q is appropriately defined, it seems this prediction equation provides values within 1 $W \cdot kg^{-1}$ of gold 465 466 standard measures (Pavei et al., 2019), however more work is needed to describe how q varies 467 during deceleration and change of direction at varied intensities. Until these data are available it 468 seems reasonable to fix q between 0.1 and 0.2 for intermittent running bouts.

469

470 4) The model is presently described to apply to an environmental state where there is strictly 'no 471 wind' (equation 11). As such the additional mechanical demand of overcoming a head-wind 472 (added resistive force) or reduced mechanical demand in the presence of a tail-wind is not 473 considered. Where wind direction and speed are able to be measured, equation 11 can be modified to accommodate these effects. Using the participant characteristics in this analysis, a 5 m \cdot s⁻¹ head 474 wind when running at 10 m·s⁻¹ increases mechanical power by 1.37 W·kg⁻¹, reducing to just 0.15 475 $W \cdot kg^{-1}$ when running at 3 m·s⁻¹. The practical significance of this assumption is therefore context 476 477 specific.

478

5) The mechanical work done to ventilate, circulate blood and other functions within the trunk andlimbs is not accounted for, which is often the case in biomechanical modelling.

481 Practical Implications

482 Gray et al. (2018) recently proposed temporal classification of movement events e.g. walking 483 bouts, running bouts, contact events etc. and subsequent energy-based quantification of these 484 movement events in field-based games. The model presented and evaluated is proposed as a 485 method to quantify the mechanical demands of identified running events. The present analyses 486 have demonstrated how the model serves to account for the demands of constant low- and high-487 speed running events, acceleration events and deceleration events, so that applied researchers and practitioners understand how global load metrics such as mechanical work done (J·kg⁻¹) in a 488 489 running based session may be derived; in this case, from the well described relationships between 490 running velocity and running kinematics (Gray et al., In Press; Pavei et al., 2019; Saibene &
491 Minetti, 2003).

492

493 Users applying the model must remain cognisant of the assumptions outlined previously. The 494 authors readily acknowledge these limitations and consider the model to provide reasonable 495 estimates of mechanical demand and power outside a laboratory setting. Work estimates produced 496 by the model are also subject to the quality of velocity-time data from which it is based. As such 497 users, must also familiarise themselves with the validity and reliability of commercial GPS 498 receivers and data collection factors that impact data quality (Scott, Scott & Kelly, 2016). 499 Furthermore, general application of the model to entire GPS field-sport match files is not 500 appropriate, as the model assumes forward running is the only gait adopted. Separate models 501 should be used to discretely evaluate other gaits and match events (Gray et al., 2018).

502

503 Given the proposed application of the model, and the low mechanical demand attributable to air 504 resistance during running (Pugh, 1971, 1976), the importance of including air resistance as a load 505 during team-sport training and competition, is questionable. Particularly, as players spend a majority of time during team sport match play at low speeds (i.e. $< 3 \text{ m} \cdot \text{s}^{-1}$) (Bangsbo et al., 2006; 506 507 Duthie et al., 2003; Gray & Jenkins, 2010), where air resistance is negligible (Figure 2). As such 508 the authors note that whilst the inclusion of W_{air} provides a more complete description, its inclusion 509 in applied practice may not be necessitated. Indeed, others readily omit this component (di 510 Prampero, Botter & Osgnach, 2015) to simplify the analysis.

512 Conclusions

513 This study presents a new approach to quantify the mechanical demands of intermittent running, 514 as measured using GPS technology. The running model presented and evaluated is proposed as 515 part of a broader energy-based solution to the quantification of field sport match demands via 516 micro-technology (Gray et al., 2018). The model uses established relationships between forward 517 running velocity and running kinematics to model the work done during a running bout. Whilst 518 this is based on several assumptions, the model provides reasonable approximations of mechanical 519 demand and power, that are responsive to varied running patterns, as evidenced in this analysis. 520 The present model may be considered an initial step toward achieving an optimal energy-based 521 method of quantifying load through micro-technology. Indeed, many attributes of this model could 522 be refined and improved upon through direct measurement rather than prediction e.g. stride 523 frequency, and/or experimental work to improve various components e.g. Wlimbs. Modelled 524 mechanical power during extended overground running may also open new avenues for research 525 and possibly strengthen our understanding of running performance, just as power-based concepts 526 have done for cycling (Shearman, Dwyer, Skiba & Townsend, 2016; Waldron, Gray, Furlan & 527 Murphy, 2016).

528

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533

534 Declaration of Interest Statement

5 The authors report no conflict of interest

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706	Tables
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736 Figure Captions737

Fig 1. Velocity curves from the fastest (long dashed line) and slowest (short dashed line) participants' 40 m sprints. The solid line is the exponential model $v_t = 9.16 \times \left(1 - e^{\frac{-t}{1.4}}\right)$, which approximates the group's sprint performance, where v is in m·s⁻¹ and t is in s.

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Fig 2. The modelled mechanical demand (*D*) i.e. work done per unit distance to a) raise and lower the COM (D_{vert}^+ and D_{vert}^- combined); b) overcome air resistance; and c) swing the limbs during constant velocity, overground running. Note: As horizontal acceleration and deceleration are zero, no horizontal work is done; therefore, D_{hor}^+ and D_{hor}^- are not included.

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Fig 3. The modelled total mechanical demand (D_{total}) i.e. work done per unit distance during constant velocity, overground running. This relationship is well described by the 4th order polynomial: $D_{total} = 0.0015v^4 - 0.0384v^3 + 0.4282v^2 - 1.975v + 4.7003$, where D_{total} is in J·kg⁻¹·m⁻¹ and *v* is in m·s⁻¹.

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Fig 4. The modelled relative contributions (%) of P_{hor} (solid line), P_{vert} (dotted line), P_{air} (long dashed line) and P_{limbs} (short dashed line) to P_{total} during constant velocity, overground running.

Fig 5. A kinematic and energetic description of a simulated 40 m sprint, including a) the velocitytime curve; b) the time-course of the modelled total mechanical demand (D_{total}); and c) the timecourse of the modelled mechanical power (P_{total}) of the running bout.

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Fig 6. A kinematic and energetic description of a hard, voluntary deceleration performed by Participant 6, including a) the velocity-time curve; b) the time-course of the modelled total mechanical demand (D_{total}); and c) the time-course of the modelled mechanical power (P_{total}) of the running bout.

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Fig 7. The time-course of the mechanical power curves for P_{hor} , P_{vert} , P_{air} and P_{limbs} during the simulated 40 m sprint [panels a), b), c) and d), respectively]; and the hard, voluntary deceleration

- 766 performed by Participant 6 [panels e), f), g) and h), respectively]. Note: the peak acceleration
- during the 40 m sprint was 5.7 m s⁻², whilst the peak deceleration by Participant 6 was -6.6 m s⁻².