

626. Higher-order approximation of cubic–quintic duffing model

S. S. Ganji ^a, A. Barari ^{b*}, H. Babazadeh ^c, M. G. Sfahani ^d, G. Domairry ^d

^a Department of Civil and Transportation Engineering, Islamic Azad University, Science and Research Branch, Iran

^b Department of Civil Engineering, Aalborg University, Sohngårdsholmsvej 57, DK-9000 Aalborg, Aalborg, Denmark

^c Department of Electrical Engineering, California State University, Los Angeles, USA

^d Departments of Civil and Mechanical Engineering, Babol University of Technology, P. O. Box 484, Babol, Iran

E-mail: * ab@civil.aau.dk, amin78404@yahoo.com

(Received 11 February 2011; accepted 15 May 2011)

Abstract. We apply an Artificial Parameter Lindstedt-Poincaré Method (APL-PM) to find improved approximate solutions for strongly nonlinear Duffing oscillators with cubic–quintic nonlinear restoring force. This approach yields simple linear algebraic equations instead of nonlinear algebraic equations without analytical solution which makes it a unique solution. It is demonstrated that this method works very well for the whole range of parameters in the case of the cubic–quintic oscillator, and excellent agreement of the approximate frequencies with the exact one has been observed and discussed. Moreover, it is not limited to the small parameter such as in the classical perturbation method. Interestingly, This study revealed that the relative error percentage in the second-order approximate analytical period is less than 0.042% for the whole parameter values. In addition, we compared this analytical solution with the Newton–Harmonic Balancing Approach. Results indicate that this technique is very effective and convenient for solving conservative truly nonlinear oscillatory systems. Utter simplicity of the solution procedure confirms that this method can be easily extended to other kinds of nonlinear evolution equations.

Keywords: Artificial Parameter Lindstedt-Poincaré Method (APL-PM); Nonlinear Cubic-Quintic Oscillation, Duffing equation.

1. Introduction

Since most phenomena in our world are essentially nonlinear and are described by nonlinear equations, the study of nonlinear vibrations and oscillations is of crucial importance in all areas of engineering sciences. Therefore, the investigation of approximate solutions of nonlinear equations can play an important role in the study of nonlinear physical phenomena. Recently, many analytical and numerical methods have emerged for solving complicated nonlinear systems. Some of these problems which are related to the cubic–quintic Duffing equation include: the nonlinear dynamics of a beam on an elastic substrate [1], the generalized Pochhammer–Chree (PC) equations [2], and the compound Korteweg–de Vries (KdV) equation [3] in nonlinear wave systems and the propagation of a short electromagnetic pulse in a nonlinear medium [4]. More recently, many effective methods [5–55] have been presented to solve these complicated nonlinear oscillation systems including: Homotopy Perturbation [6–10], Parameter-Expanding (Expansion) [11], Multiple Scale [12–14], Harmonic Balance and

Duffing Harmonic Balance [15-18], Incremental Harmonic Balance [19, 20], Variational Iteration [21-22], Variational Approach [23–26], Max-Min [27, 28], Amplitude-Frequency Formulation [29], Linearized Perturbation [30], Energy Balance [25, 31–32], Power Series [33], homotopy analysis [34], Finite Element [35], Iteration Procedures [36, 37], Newton–Harmonic Balancing [38], Lindstedt–Poincaré [39, 40], Improved Lindstedt–Poincaré [41, 42], as well as other powerful methods which are available in the literature [43-52].

Ramos [53-55] proposed an Artificial Parameter Lindstedt–Poincaré Method (APL-PM), to obtain periodic solutions. Applications of this method can be found in [53-55] for solving nonlinear evolution equations arising in mathematical fields.

The main motivation of the present work is to extend the APL-PM to a generalized cubic–quintic Duffing with variable coefficients.

2. Artificial Parameter Lindstedt–Poincaré Method

Because of this fact that many important equations raised in practical engineering systems [1, 4] are in the form of Duffing equation, it seems to be more fundamental to consider equations presented in the following general form:

$$\frac{d^2u}{dt} + f(u) = 0 \tag{1}$$

With initial conditions

$$u(0) = A, \quad \frac{du}{dt}(0) = 0 \tag{2}$$

Where $f(u)$ is an odd function, and u and t are generalized dimensionless displacement and time variables. By defining a new independent variable replacing the time variable, $t = \theta/\omega$, Eq. (1) can be written as [53]:

$$\omega^2 u'' + u = p[u - f(u)], \quad u(0) = A, \quad u'(0) = 0 \tag{3}$$

Eq. (3) coincides with Eq. (2) for $p = 1$. Applying Artificial Parameter Lindstedt–Poincaré procedure, the displacement and angular frequency can be expressed as Eqs. (5) and (6), respectively:

$$u = u_0 + pu_1 + p^2u_2 + \dots, \tag{4}$$

$$\omega = \sqrt{\omega_0^2 + p\omega_1^2 + p^2\omega_2^2 + \dots} \tag{5}$$

Substituting Eqs. (4) and (5) into Eq. (3) results in:

$$\begin{aligned} & \left(\omega_0^2 + \sum_{i=1}^{+\infty} p^i \omega_i^2 \right) \left(u_0'' + \sum_{i=1}^{+\infty} p^i u_i'' \right) + \left(u_0^2 + \sum_{i=1}^{+\infty} p^i u_i^2 \right) \\ & = p \left[\left(u_0^2 + \sum_{i=1}^{+\infty} p^i u_i^2 \right) - f \left(u_0^2 + \sum_{i=1}^{+\infty} p^i u_i^2 \right) \right] \end{aligned} \tag{6}$$

Expanding Eqs. (6), gives:

$$p^0 : u_0'' + \omega_0^2 u_0 = 0, \tag{7}$$

$$p^1 : u_1'' + \omega_1^2 u_1 - K(u_0, u_0', u_0'', \omega_0, \omega_1) = 0, \tag{8}$$

$$p^2 : u_2'' + \omega_2^2 u_2 - G(u_i, u_i', u_i'', \omega_j) = 0, \quad i = 0, 1, \quad j = 0, 1, 2, \tag{9}$$

⋮

Where $K(u_0, u_0', u_0'', \omega_j)$ and $G(u_i, u_i', u_i'', \omega_j)$ are linear differential term. These equations can be solved stage by stage. For the primary stage, the solution of Eq. (7) is:

$$u_0 = A \cos(\theta) \tag{10}$$

By substituting Eq. (10) into Eq. (8):

$$p^1 : u_1'' + \omega_1^2 u_1 - K(A \cos(\theta), -A \sin(\theta), -A \cos(\theta), \omega_0, \omega_1) = 0 \tag{11}$$

Using Fourier expansion series, we can rewrite the right hand of Eq. (11) in the following form:

$$\begin{aligned} &K(A \cos(\theta), -A \sin(\theta), -A \cos(\theta), \omega_0, \omega_1) \\ &= \sum_{n=0}^{\infty} b_{2n+1} \cos[(2n+1)\omega t] = \sum_{n=0}^{\infty} b_{2n+1} \cos[(2n+1)\omega t] \approx b_1 \cos(\omega t) \end{aligned} \tag{12}$$

By setting $b_1 = 0$ and solving it, we can achieve ω_0 . The solution of Eq. (7) using ω_0 gives u_1 . Then, solving Eq. (8) with ω_0 and u_1 yields ω_1 . These stages can continue for better results.

3. Implementation of APLPM to cubic–quintic Duffing model

We governed the cubic–quintic Duffing model by a nonlinear differential equation with all real and positive coefficients. In this regard, the general form of cubic–quintic Duffing equation, Eq. (13), is considered.

$$\frac{du}{dt} + \alpha u + \beta u^3 + \gamma u^5 = 0, \quad u(0) = A, \quad \frac{du}{dt}(0) = 0 \tag{13}$$

In order to use the APLPM, by applying new variable, $t = \theta/\omega$, we have:

$$\omega^2 u'' + u = p \left([1 - \alpha]u - \beta u^3 - \gamma u^5 \right) = 0, \quad u(0) = A, \quad u'(0) = 0 \tag{14}$$

Where $u'' = du/d\theta$. Substituting Eqs. (5) and (6) into Eq. (14) and equating the terms with the identical powers of p , yields:

$$p^0 : u_0'' + u_0 = 0, \tag{15}$$

$$p^1 : u_1'' + u_1 = -\frac{\omega_1^2}{\omega_0^2} u_0'' - \frac{\omega_1^2}{\omega_0^2} u_0 - \frac{\gamma}{\omega_0^2} u_0^5 - \frac{\beta}{\omega_0^2} u_0^3 - \frac{\alpha}{\omega_0^2} u_0 + u_0, \tag{16}$$

$$\begin{aligned}
 p^2 : & \quad u_2'' + u_2 \\
 = & \quad -\frac{5\gamma u_0^4 u_1}{\omega_0^2} - \frac{\omega_1^2 u_0}{\omega_0^2} + u_1 - \frac{3\beta u_0^2 u_1}{\omega_0^2} - \frac{\omega_1^2 u_1''}{\omega_0^2} + \frac{\omega_1^2 u_0}{\omega_0^2} - \frac{\omega_2^2 u_0''}{\omega_0^2} - \frac{\omega_1^2 u_1}{\omega_0^2} - \frac{\alpha u_1}{\omega_0^2}, \quad (17)
 \end{aligned}$$

⋮

3.1. First-order analytical approximation

At this step, we solve Eq. (15) with the initial values $u(0) = A$ and $u'(0) = 0$, which is leads to $u_0 = A \cos \theta$. Substituting u_0 into Eq. (16) and simplifying the result, we obtain:

$$u_1'' + u_1 = A \cos t - \frac{1}{\omega_0^2} (\alpha A \cos \theta + \beta A^3 \cos^3 \theta + \gamma A^5 \cos^5 \theta) \quad (18)$$

It is possible to apply the following Fourier series expansion:

$$u_1'' + u_1 = (A + b_1) \cos \theta + \sum_{n=1}^{\infty} b_{2n+1} \cos [(2n + 1)\theta] \quad (19)$$

where b_1 is as follow:

$$\begin{aligned}
 b_1 &= \frac{4}{\pi \omega_0^2} \cdot \int_0^{\pi/2} (\cos \varphi [\alpha A \cos \varphi + \beta A^3 \cos^3 \varphi + \gamma A^5 \cos^5 \varphi]) d\varphi \\
 &= -\frac{A(18A^2\beta + 24\alpha + 15A^4\gamma)}{24\omega_0^2} \quad (20)
 \end{aligned}$$

Substituting Eq. (20) into Eq. (19) gives:

$$\begin{aligned}
 u_1'' + u_1 &= \left(\frac{24\omega_0^2 A - A(18A^2\beta + 24\alpha + 15A^4\gamma)}{24\omega_0^2} \right) \cos \theta \\
 &+ \sum_{n=1}^{\infty} b_{2n+1} \cos [(2n + 1)\theta] \quad (21)
 \end{aligned}$$

No secular term in u_1 requires that:

$$\frac{24\omega_0^2 A - A(18A^2\beta + 24\alpha + 15A^4\gamma)}{24\omega_0^2} = 0 \quad (22)$$

Solving Eq. (22), we obtain the first order approximate solution of Eq. (1) as follow:

$$\omega_{1th} = \omega_0 = \sqrt{\alpha + \frac{3\beta A^2}{4} + \frac{5\gamma A^4}{8}}, \quad (23)$$

Where angular frequency ω_0 is the first-order analytical approximation. Eq. (23) gives the same frequency resulted in by the applications of the harmonic balance and the first order approximation of Newton–harmonic balancing approach [38]. Therefore, the corresponding

approximate analytical periodic solution u_1 can then be achieved by substituting Eq. (23) into Eq. (16) as:

$$u_1 = -\frac{A^3 \sin^2 \theta \cos \theta (2\gamma A^2 \cos^2 \theta + 6\beta + 7\gamma A^2)}{6(8\alpha + 6\beta A^2 - 5\gamma A^4)} \quad (24)$$

3.1. Second-order analytical approximation

To determine the second-order approximate solution, it is necessary to substitute $u_0 = A \cos \theta$ and Eqs. (23) and (24) into Eq. (17). So, we can obtain:

$$u_2'' + u_2 = \frac{A\psi \cos \theta}{6(6A^2\beta + 8\alpha + 5A^4\gamma)},$$

$$\begin{aligned} \psi = & -80\gamma^2 A^8 \cos^8 \theta - 200\gamma^2 A^8 \cos^6 \theta - 288\gamma \beta A^6 \cos^6 \theta + 290\gamma^2 A^8 \cos^4 \theta + \\ & 132\gamma \beta A^6 \cos^4 \theta + 384\gamma \omega_1^2 A^4 \cos^4 \theta - 144\beta^2 A^4 \cos^4 \theta + 25\gamma^2 A^8 \cos^2 \theta + \\ & 228\gamma \beta A^6 \cos^2 \theta + 180\beta^2 A^4 \cos^2 \theta + 384\beta \omega_1^2 A^2 \cos^2 \theta - 35\gamma^2 A^8 - 72\gamma \beta A^6 + \\ & -36A^4 \beta^2 + 384\alpha \omega_1^2 \end{aligned} \quad (25)$$

Similar to the first step, using the Fourier series, the right hand of Eq. (25) will become as:

$$\begin{aligned} A\psi \cos \theta &= \sum_{n=0}^{\infty} b_{2n+1} \cos[(2n+1)\theta] = \left(\frac{4}{\pi} \int_0^{\pi/2} (\psi \cos^2 \varphi) d\varphi \right) \cos \theta \\ &+ \sum_{n=1}^{\infty} b_{2n+1} \cos[(2n+1)\theta] \\ &= \left[0.25A \left(\begin{aligned} &65A^8\gamma^2 + 36A^4\beta^2 + 1536\alpha\omega_1^2 \\ &+ 96A^6\beta\gamma + 960A^4\gamma\omega_1^2 + 1152\omega_1^2 A^2\beta \end{aligned} \right) \right] \cos(\theta) \\ &+ \sum_{n=1}^{\infty} b_{2n+1} \cos[(2n+1)\theta] \end{aligned} \quad (26)$$

From Eqs. (25) and (26) we can obtain the secular term as:

$$\left[0.25A \left(\begin{aligned} &65A^8\gamma^2 + 36A^4\beta^2 + 1536\alpha\omega_1^2 \\ &+ 96A^6\beta\gamma + 960A^4\gamma\omega_1^2 + 1152\omega_1^2 A^2\beta \end{aligned} \right) \right] \quad (27)$$

No secular term in u_2 requires that:

$$\omega_1 = \frac{A^2 \sqrt{-(18A^2\beta - 24\alpha + 15\gamma A^4)(65\gamma^2 A^4 + 96A^2\beta\gamma + 36\beta^2)}}{24(6A^2\beta + 8\alpha + 5A^4\gamma)^2} \quad (28)$$

From Eqs. (23 and 28) into Eq. (5), we can obtain the angular frequency ω_{2th} as a more accurate and higher-order approximation compared to ω_0 :

$$\omega_{2th} = \sqrt{\omega_0^2 + \alpha_1^2} \tag{29}$$

4. Illustrative examples of cubic–quintic Duffing oscillators

In order to verify the effectiveness of the proposed higher-order analytical approximate method, the exact solutions and another approximate solution, namely, Newton–harmonic balancing approach is used to compute the angular frequencies of the cubic–quintic oscillator for different parameters. The various approximate solutions are presented in Figs. 1–4. Tables 1–4 list the corresponding numerical results of angular frequencies. These figures and tables correspond to small and large amplitudes of oscillation for different parameters of α , β and γ . For reference, the exact frequency ω_e is obtained by direct integration of governing nonlinear differential Eq. (1) of the dynamical system. Imposing the initial conditions, the solution is [38]:

$$\omega_e(A) = \frac{\pi k_1}{2 \int_0^{\pi/2} (1 + k_2 \sin^2 t + k_3 \sin^4 t)^{-1/2} dt},$$

$$k_1 = \sqrt{\alpha + \frac{\beta A^2}{2} + \frac{\gamma A^4}{3}},$$

$$k_2 = \frac{3\beta A^2 + 2\gamma A^4}{6\alpha + 3\beta A^2 + 2\gamma A^4},$$

$$k_3 = \frac{2\gamma A^4}{6\alpha + 3\beta A^2 + 2\gamma A^4}.$$
(30)

The approach presented herein is suitable for small as well as large amplitudes of oscillation. The various limiting approximations for $A \rightarrow \infty$ can be derived based on Eqs. (23) and (29) as follows:

$$\lim_{A \rightarrow \infty} \frac{\omega_{1th}(A)}{\omega_e(A)} = \lim_{A \rightarrow \infty} \frac{T_{1th}(A)}{T_e(A)} = 1.05856 \tag{31}$$

$$\lim_{A \rightarrow \infty} \frac{\omega_{2th}(A)}{\omega_e(A)} = \lim_{A \rightarrow \infty} \frac{T_{2th}(A)}{T_e(A)} = 0.99958 \tag{32}$$

From Eqs. (31) and (32), it is obvious that the relative errors of the first-order and second-order approximations of APL-PM as compared to the exact solution are less than 5.86 % and 0.042 %, respectively. It is noted that the maximum errors of the third-order approximation of Newton–harmonic balancing approach to the exact solution is 0.23. As it can be seen through Tables 1-4, the first-order solutions of two approximate methods are equal and relatively inaccurate. Although the third-order solution of NHBA is good, the second-order solution of APL-PM is excellent.

The frequency ratio and amplitude relationship of the cubic–quintic Duffing oscillator governed by $u'' + u + u^3 + u^5 = 0$ is shown in Fig. 1. As observed, the second-order approximation of the proposed method has satisfactory agreement with the exact solution for $A \in [0.1, 1000]$. Figs. (2-4) are also related to, $u'' + 2u + u^3 + u^5 = 0$, $u'' + 5u + 3u^3 + u^5 = 0$ and $u'' + u + 10u^3 + 100u^5 = 0$, respectively.

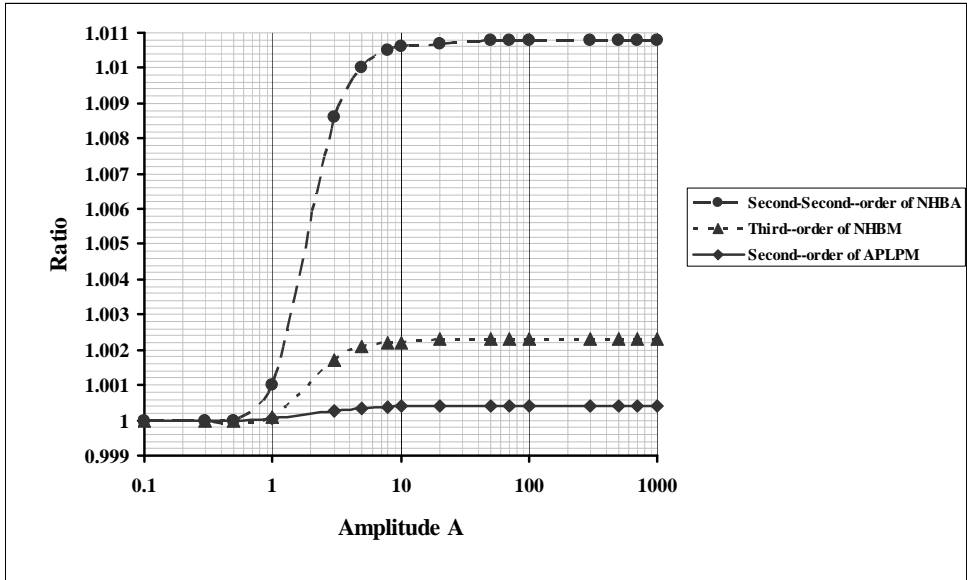


Fig. 1. Comparison of approximate frequencies with the exact frequency for cubic–quintic Duffing oscillator for $\alpha = \beta = \gamma = 1$

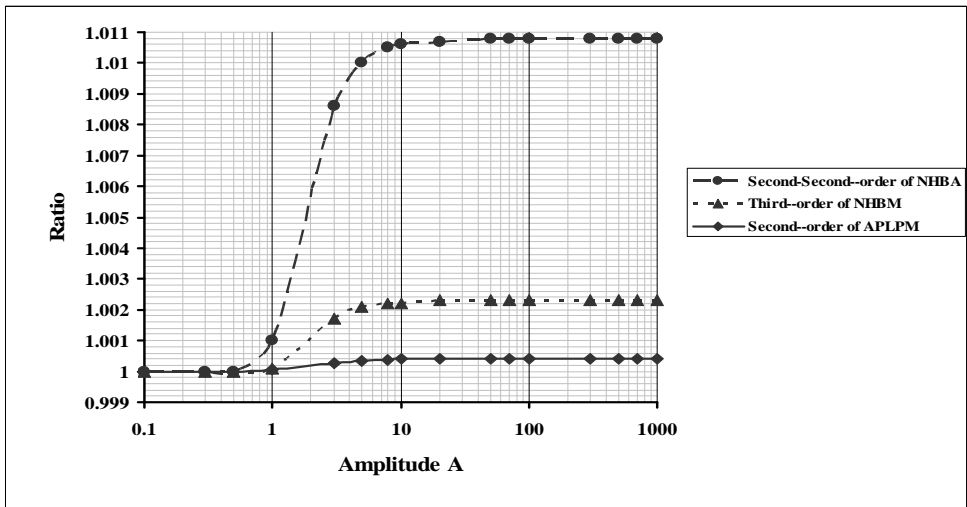


Fig. 2. Comparison of approximate frequencies with exact frequency for the cubic–quintic Duffing oscillator for $\alpha = 2$ and $\beta = \gamma = 1$

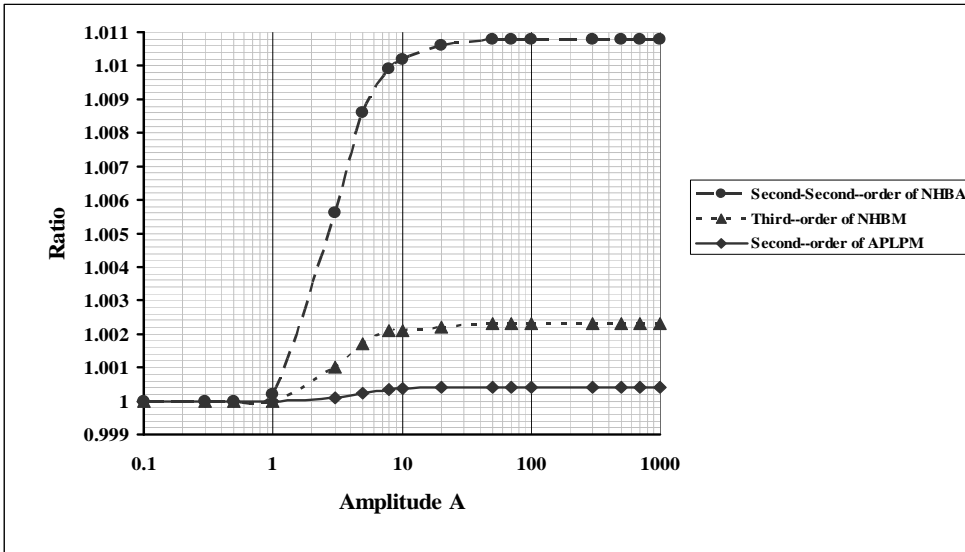


Fig. 3. Comparison of approximate frequencies with the exact frequency for cubic–quintic Duffing oscillator for $\alpha = 5$, $\beta = 3$ and $\gamma = 1$

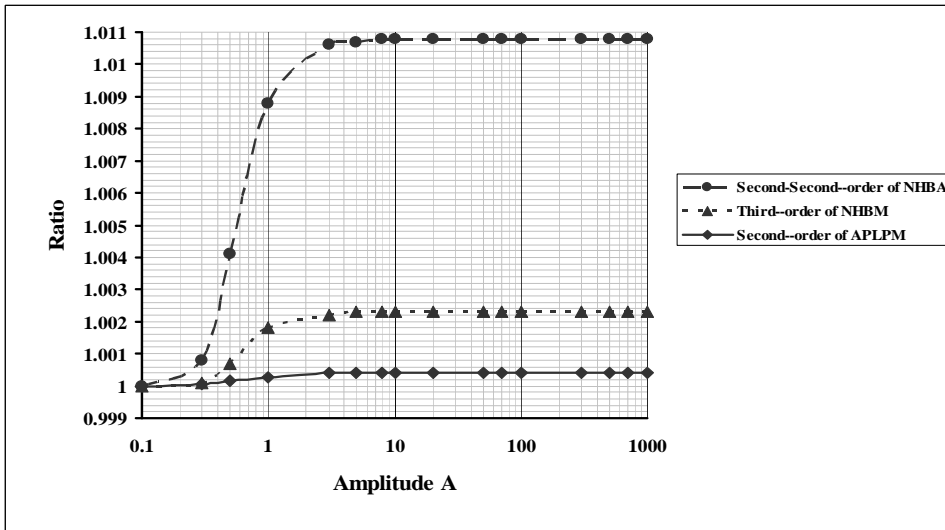


Fig. 4. Comparison of approximate frequencies with the exact frequency for cubic–quintic Duffing oscillator for $\alpha = 1$, $\beta = 10$ and $\gamma = 100$

The proposed APL-PM for the second-order analytical approximation demonstrates noticeable improvement as compared with the lower-order analytical approximation. It is also observed that the method is effective for solving highly nonlinear oscillators with cubic and/or quintic nonlinearity and it has clear advantage over the classical perturbation method, which is restricted by the presence of a small parameter in the governing differential equation.

Table 1. Percentage of errors for comparison of approximate frequencies with exact frequency for $\alpha = \beta = \gamma = 1$

A	ω_e	Newton–Harmonic Balancing Approach			Artificial Parameter Lindstedt–Poincaré	
		ω_{1th}	ω_{2th}	ω_{3th}	ω_{1th}	ω_{2th}
0.1	1.00377	1.00377 (0.00)	1.00377 (0)	1.00377 (0)	1.00377 (0.00)	1.00377 (0.00)
0.3	1.03554	1.03565 (0.01)	1.03554 (0)	1.03554 (0)	1.03565 (0.01)	1.03554 (0.00)
0.5	1.10654	1.10750 (0.09)	1.10658 (0)	1.10655 (0)	1.10750 (0.09)	1.10654 (0.00)
1	1.52359	1.54110 (1.15)	1.52507 (0.10)	1.52375 (0.01)	1.54110 (1.15)	1.52348 (0.007)
3	7.26863	7.64035 (5.11)	7.33114 (0.86)	7.28115 (0.17)	7.64035 (5.11)	7.26675 (0.026)
5	19.1815	20.2577 (5.61)	19.3735 (1.00)	19.2215 (0.21)	20.2577 (5.61)	20.2577 (0.034)
8	48.2946	51.0784 (5.76)	48.8010 (1.05)	48.4011 (0.22)	51.0784 (5.76)	51.0784 (0.039)
10	75.1774	79.5362 (5.80)	75.9738 (1.06)	75.3454 (0.22)	79.5362 (5.80)	75.1474 (0.040)
20	299.223	316.703 (5.84)	302.435 (1.07)	299.903 (0.23)	316.703 (5.84)	299.099 (0.041)
50	1867.57	1976.90 (5.85)	1887.69 (1.08)	1871.84 (0.23)	1976.90 (5.85)	1866.79 (0.042)
70	3659.98	3874.26 (5.86)	3699.42 (1.08)	3668.33 (0.23)	3874.26 (5.86)	3658.44 (0.042)
100	7468.83	7906.17 (5.86)	7549.34 (1.08)	7485.89 (0.23)	7906.17 (5.86)	7465.69 (0.042)
300	67215.57	71151.72 (5.86)	67940.22 (1.08)	67369.12 (0.23)	71151.72 (5.86)	67187.26 (0.042)
500	186709.04	197642.83 (5.86)	188721.99 (1.08)	187135.59 (0.23)	197642.83 (5.86)	186630.42 (0.042)
700	365949.25	387379.49 (5.86)	369894.64 (1.08)	366785.29 (0.23)	387379.49 (5.86)	365795.14 (0.042)
1000	746834.69	790569.89 (5.86)	754886.52 (1.08)	748540.91 (0.23)	790569.89 (5.86)	746520.19 (0.042)

Table 2. Percentage of errors for comparison of approximate frequencies with exact frequency for $\alpha = 2$ and $\beta = \gamma = 1$

A	ω_e	Newton–Harmonic Balancing Approach			Artificial Parameter Lindstedt–Poincaré	
		ω_{1th}	ω_{2th}	ω_{3th}	ω_{1th}	ω_{2th}
0.1	1.41688	1.41688 (0.00)	1.41688 (0.00)	1.41688 (0.00)	1.41688 (0.00)	1.41688 (0.00)
0.3	1.43960	1.43964 (0.003)	1.43960 (0.00)	1.43960 (0.00)	1.43964 (0.003)	1.43960 (0.00)
0.5	1.49177	1.49217 (0.03)	1.49179 (0.00)	1.49178 (0.00)	1.49217 (0.03)	1.49177 (0.00)
1	1.82682	1.83712 (0.56)	1.82763 (0.04)	1.82689 (0.004)	1.83712 (0.56)	1.82674 (0.004)
3	7.34386	7.70552 (4.92)	7.40403 (0.82)	7.35578 (0.16)	7.70552 (4.92)	7.34163 (0.030)
5	19.2104	20.2824 (5.58)	19.4014 (0.99)	19.2501 (0.21)	20.2824 (5.58)	19.2037 (0.039)
8	48.3061	51.0882 (5.76)	48.8120 (1.05)	48.4125 (0.22)	51.0882 (5.76)	48.2873 (0.040)
10	75.1848	79.5424 (5.80)	75.9809 (1.06)	75.3527 (0.22)	79.5424 (5.80)	75.1548 (0.040)
20	299.225	316.705 (5.84)	302.437 (1.07)	299.905 (0.23)	316.705 (5.84)	299.101 (0.041)
50	1867.57	1976.90 (5.85)	1887.70 (1.08)	1871.84 (0.23)	1976.90 (5.85)	1866.79 (0.042)
70	3659.98	3874.26 (5.85)	3699.42 (1.08)	3668.33 (0.23)	3874.26 (5.85)	3658.44 (0.042)
100	7468.83	7906.17 (5.86)	7549.34 (1.08)	7485.89 (0.23)	7906.17 (5.86)	7465.69 (0.042)
300	67215.57	71151.72 (5.86)	67940.22 (1.08)	67369.12 (0.23)	71151.72 (5.86)	67187.26 (0.042)
500	186709.04	197642.83 (5.86)	188721.99 (1.08)	187135.59 (0.23)	197642.83 (5.86)	186630.42 (0.042)
700	365949.25	387379.49 (5.86)	369894.64 (1.08)	366785.29 (0.23)	387379.49 (5.86)	365795.14 (0.042)
1000	746834.69	790569.89 (5.86)	754886.52 (1.08)	748540.91 (0.23)	790569.89 (5.86)	746520.19 (0.042)

5. Conclusion

The Artificial Parameter Lindstedt–Poincaré method has been applied to obtain periodic solution for truly cubic–quintic nonlinear oscillator. The major conclusion is that this approach provides excellent approximation to the solution of these nonlinear systems with high accuracy for the whole solution domain. The analytical representations obtained using the Artificial Parameter Lindstedt–Poincaré technique give excellent approximations to the exact solutions for the whole range of amplitude values.

These approximate solutions are better than the approximate solutions obtained using the Newton–harmonic balancing approach. For the second order approximation, the maximum

relative error of the analytical approximate frequency obtained using the APL–PM for the cubic–quintic oscillator is 0.04%, while the relative error is 1.08% when the second order approximation is considered for the Newton–harmonic balancing approach. An interesting feature considered in this paper is to show that the second order approximation result of APL–PM is better than the third order approximation of Newton–harmonic balancing approach, which constitutes 0.23%.

Table 3. Percentage of errors for comparison of approximate frequencies with exact frequency for $\alpha = 5$, $\beta = 3$ and $\gamma = 1$

<i>A</i>	ω_e	Newton–Harmonic Balancing Approach			Artificial Parameter Lindstedt–Poincaré	
		ω_{1th}	ω_{2th}	ω_{3th}	ω_{1th}	ω_{2th}
0.1	2.24111	2.24111 (0.00)	2.24111 (0.00)	2.24111 (0.00)	2.24111 (0.00)	2.24111 (0.00)
0.3	2.28193	2.28201 (0.004)	2.28193 (0.00)	2.28193 (0.00)	2.28201 (0.004)	2.28193 (0.00)
0.5	2.36615	2.36676 (0.03)	2.36616 (0.00)	2.36615 (0.00)	2.36676 (0.03)	2.36615 (0.00)
1	2.79627	2.80624 (0.36)	2.79670 (0.02)	2.79630 (0.00)	2.80624 (0.36)	2.79625 (0.00)
3	8.37877	8.71063 (3.96)	8.42601 (0.56)	8.38730 (0.10)	8.71063 (3.96)	8.37789 (0.01)
5	20.2164	21.2574 (5.15)	20.3911 (0.86)	20.2514 (0.17)	21.2574 (5.15)	20.2118 (0.023)
8	49.2955	52.0481 (5.58)	49.7844 (0.99)	49.3969 (0.21)	52.0481 (5.58)	49.2794 (0.033)
10	76.1698	80.4984 (5.68)	76.9487 (1.02)	76.3326 (0.21)	80.4984 (5.68)	76.1425 (0.036)
20	300.204	317.655 (5.81)	303.399 (1.06)	300.879 (0.22)	317.655 (5.81)	300.083 (0.040)
50	1868.55	1977.85 (5.85)	1888.65 (1.08)	1872.81 (0.23)	1977.85 (5.85)	1867.77 (0.042)
70	3660.95	3875.21 (5.85)	3700.38 (1.08)	3669.31 (0.23)	3875.21 (5.85)	3659.42 (0.042)
100	7469.81	7907.12 (5.85)	7550.30 (1.08)	7486.86 (0.23)	7907.12 (5.85)	7466.66 (0.042)
300	67216.54	71152.67 (5.86)	67941.18 (1.08)	67370.09 (0.23)	71152.67 (5.86)	67188.24 (0.042)
500	186710.01	197643.78 (5.86)	188722.95 (1.08)	187136.56 (0.23)	197643.78 (5.86)	186631.40 (0.042)
700	365950.22	387380.44 (5.86)	369895.60 (1.08)	366786.26 (0.23)	387380.44 (5.86)	365796.12 (0.042)
1000	746835.66	790570.84 (5.86)	754887.48 (1.08)	748541.88 (0.23)	790570.84 (5.86)	746521.17 (0.042)

Table 4. Percentage of errors for comparison of approximate frequencies with exact frequency for $\alpha = 1$, $\beta = 10$ and $\gamma = 100$

<i>A</i>	ω_e	Newton–Harmonic Balancing Approach			Artificial Parameter Lindstedt–Poincaré	
		ω_{1th}	ω_{2th}	ω_{3th}	ω_{1th}	ω_{2th}
0.1	1.03970	1.03983 (0.01)	1.03970 (0.00)	1.03970 (0.00)	1.03983 (0.01)	1.03970 (0.00)
0.3	1.46259	1.47691 (0.98)	1.46373 (0.08)	1.46271 (0.01)	1.47691 (0.98)	1.46251 (0.006)
0.5	2.52469	2.60408 (3.14)	2.53505 (0.41)	2.52642 (0.07)	2.60408 (3.14)	2.52426 (0.017)
1	8.01005	8.42615 (5.19)	8.08069 (0.88)	8.02429 (0.18)	8.42615 (5.19)	8.00790 (0.027)
3	67.7097	71.6310 (5.79)	68.4255 (1.06)	67.8606 (0.22)	71.6310 (5.79)	67.6828 (0.040)
5	187.199	198.119 (5.83)	189.203 (1.07)	187.623 (0.23)	198.119 (5.83)	187.122 (0.041)
8	478.463	506.440 (5.85)	483.608 (1.08)	479.552 (0.23)	506.440 (5.85)	478.263 (0.042)
10	747.323	791.044 (5.85)	755.366 (1.08)	749.027 (0.23)	791.044 (5.85)	747.010 (0.042)
20	2987.83	3162.75 (5.85)	3020.02 (1.08)	2994.65 (0.23)	3162.75 (5.85)	2986.57 (0.042)
50	18671.34	19764.71 (5.86)	18872.63 (1.08)	18714.00 (0.23)	19764.71 (5.86)	18663.48 (0.042)
70	36595.36	38738.38 (5.86)	36989.90 (1.08)	36678.97 (0.23)	38738.38 (5.86)	36579.95 (0.042)
100	74683.91	79057.42 (5.86)	75489.08 (1.08)	74854.53 (0.23)	79057.42 (5.86)	74652.46 (0.042)
300	672151.27	711512.95 (5.86)	679397.92 (1.08)	673686.86 (0.23)	711512.95 (5.86)	671868.22 (0.042)
500	1867085.99	1976424.01 (5.86)	1887215.59 (1.08)	1871351.54 (0.23)	1976424.01 (5.86)	1866299.75 (0.042)
700	3659488.07	3873790.61 (5.86)	3698942.10 (1.08)	3667848.54 (0.23)	3873790.61 (5.86)	3657947.03 (0.042)
1000	7468342.49	7905694.62 (5.86)	7548860.93 (1.08)	7485404.69 (0.23)	7905694.62 (5.86)	7465197.52 (0.042)

In general, the first-order periodic solution of APL–PM is generally acceptable as compared to the exact solution while the second-order periodic solution is in good and excellent agreement with the exact solution. In summary, proposed method is simple in its principle, and can be used to solve other conservative truly nonlinear oscillators with complex nonlinearities.

References

- [1] **S. Lenzi, G. Menditto, A. M. Tarantino.** Homoclinic and heteroclinic bifurcations in the non-linear dynamics of a beam resting on an elastic substrate, *Int. J. Non-Linear Mech.* 34 (1999) 615.
- [2] **Z. Yan.** A new sine–Gordon equation expansion algorithm to investigate some special nonlinear differential equations, *Chaos Soliton Fract* 23 (2005) 767.
- [3] **D. J. Huang, H. Q. Zhang.** Link between travelling waves and first order nonlinear ordinary differential equation with a sixth-degree nonlinear term, *Chaos Soliton Fract* 29 (2006) 928.
- [4] **A. I. Maimistov.** Propagation of an ultimately short electromagnetic pulse in a nonlinear medium described by the fifth-order Duffing model, *Opt. Spectrosc.* 94 (2003) 251.
- [5] **Y. K. Cheung, S. H. Chen, S. L. Lau.** Application of the incremental harmonic balance method to cubic non-linearity systems, *J. Sound Vib.* 140 (1990) 273.
- [6] **J. H. He.** An elementary introduction to recently developed asymptotic methods and nanomechanics in textile engineering, *International Journal of Modern Physics B*, 22 (21) (2008) 3487.
- [7] **A. Barari, M. Omidvar, Abdoul R. Ghotbi, D. D. Ganji.** 2008, Application of homotopy perturbation method and variational iteration method to nonlinear oscillator differential equations, *Acta Applicanda Mathematicae*, 104(2),161-171.
- [8] **M. G. Sfahani, S. S. Ganji, A. Barari, H. Mirgolbabae, G. Domairry.** 2010, Analytical Solutions to Nonlinear Conservative Oscillator with Fifth-Order Non-linearity, *Journal of Earthquake Engineering and Engineering Vibration*, 9(3), 367-374.
- [9] **S. R. Seyedalizadeh, G. Domairry, S. Karimpour.** An approximation of the analytical solution of the linear and nonlinear integro-differential equations by homotopy perturbation method, *Acta Applicandae Mathematicae*, doi: 10.1007/s10440-008-9261-z.
- [10] **S. S. Ganji, D. D. Ganji, S. Karimpour, H. Babazadeh.** Applications of He’s Homotopy Perturbation Method to Obtain Second-order Approximations of the Coupled Two-Degree-of-Freedom Systems, *International Journal of Nonlinear Science and Numerical Simulation*, 10(3) (2009), 303.
- [11] **A. Kimiaefar, E. Lund, O. T. Thomsen, A. Barari.** On Approximate Analytical Solutions of Nonlinear Vibrations of Inextensible Beams using Parameter-Expansion Method, *International Journal of Nonlinear Sciences and Numerical Simulation*, 11(9), 2010, 743-753.
- [12] **N. Okuizumi, K. Kimura.** Multiple time scale analysis of hysteretic systems subjected to harmonic excitation, *Journal of Sound and Vibration*, 272 (2004) 675.
- [13] **A. Marathe, A. Chatterjee.** Wave attenuation in nonlinear periodic structures using harmonic balance and multiple scales, *Journal of Sound and Vibration*, 289 (2006) 871.
- [14] **M. P. Cartmell, S. W. Ziegler, R. Khanin, D. I. M. Forehand.** Multiple scales analyses of the dynamics of weakly nonlinear mechanical systems, *Applied Mechanics Reviews*, 56 (2003) 455.
- [15] **H. P. W. Gottlieb.** Harmonic balance approach to limit cycles for nonlinear jerk equations, *Journal of Sound and Vibration*, 297 (2006) 243.
- [16] **J. F. Dunne, P. Hayward.** A split-frequency harmonic balance method for nonlinear oscillators with multi-harmonic forcing, *Journal of Sound and Vibration* 295 (2006) 939.
- [17] **H. Hu.** Solution of a mixed parity nonlinear oscillator: Harmonic balance, *Journal of Sound and Vibration* 299 (2007) 331.
- [18] **H. Hu, J. H. Tang.** Solution of a Duffing-harmonic oscillator by the method of harmonic balance, *Journal of Sound and Vibration* 294 (2006) 637.
- [19] **A. Y. T. Leung, S. K. Chui.** Non-linear vibration of coupled Duffing oscillators by an improved incremental harmonic balance method, *J. Sound Vib.* 181 (1995) 619–633.
- [20] **Y. K. Cheung, S. H. Chen, S. L. Lau.** A modified Lindstedt–Poincaré method for certain strongly non-linear oscillators, *Int. J. Non-Linear Mech.* 26 (1991) 367.
- [21] **A. Barari, M. Omidvar, D. D. Ganji and Abbas Tahmasebi.** 2008, An Approximate Solution for Boundary Value Problems in Structural Engineering and Fluid Mechanics, *Journal of Mathematical Problems in Engineering*, (2008), Article ID 394103, 1-13.
- [22] **M. Rafei, D. D. Ganji, H. Daniali, H. Pashaei.** The variational iteration method for nonlinear oscillators with discontinuities, *Journal of Sound and Vibration*, 305 (2007) 614.
- [23] **J. H. He.** Variational approach for nonlinear oscillators, *Chaos, Solitons and Fractals*, 34 (2007) 1430.

- [24] **J. H. He.** Variational principles for some nonlinear partial differential equations with variable coefficients, *Chaos, Solitons & Fractals*, 19 (4) (2004) 847.
- [25] **S. S. Ganji, D. D. Ganji, S. Karimpour.** He's energy balance and He's variational methods for nonlinear oscillations in engineering, *International Journal of Modern Physics B*, 23 (2009) 461.
- [26] **N. Tolou, H. Hashemi, A. Barari.** 2009, Analytical Investigation of Strongly Nonlinear Normal Mode Using HPM and He's Variational Method, *Journal of Applied Functional Analysis*, 4(1), 682-691.
- [27] **M. Ghadimi, H. D. Kaliji, A. Barari,** Analytical Solutions to Nonlinear Mechanical Oscillation Problems, *Vibroengineering*, 2011, in press.
- [28] **L. B. Ibsen, A. Barari, A. Kimiaefar.** 2010, Analysis of Strongly Nonlinear Oscillation Systems Using He's Max-Min Method and Comparison with Homotopy Analysis Method and Energy Balance, *Sadhana*, 35(4), 1-16.
- [29] **A. Fereidoon, M. Ghadimi, A. Barari, H. D. Kaliji, G. Domairry.** 2011, Nonlinear Vibration of Oscillation Systems Using Frequency-Amplitude Formulation, *Shock and Vibration*, 2011, in press.
- [30] **J. H. He,** Linearized Perturbation Technique and its applications to strongly Nonlinear Oscillators, *Computers and Mathematics with Applications*, 45 (2003) 1.
- [31] **M. G. Sfehiani, A. Barari, M. Omidvar, S. S. Ganji, G. Domairry.** 2011, Dynamic response of inextensible beams by improved energy balance method, *Proceedings of the Institution of Mechanical Engineers, Part K: Journal of Multi-body Dynamics*, 225(1), 66-73.
- [32] **M. Bayat, A. Barari, M. Shahidi.** 2011, On the Approximate Analytical Solution of Euler-Bernoulli Beams, *Mechanika*, in press.
- [33] **M. I. Qaisi.** A power series approach for the study of periodic motion, *J. Sound Vib.* 196(1996) 401.
- [34] **M. Momeni, N. Jamshidi, A. Barari, G. Domairry.** 2011, Numerical Analysis of Flow and Heat Transfer of a Viscoelastic Fluid Over A Stretching Sheet, *International Journal of Numerical Methods in Heat and Fluid Flow*, 21(2), DOI: 10.1108/09615531111105407.
- [35] **Z. G. Xiong, H. Hu.** Simplified continuous finite element method for a class of nonlinear oscillating equations, *J. Sound Vib.* 287 (2005) 367.
- [36] **H. Hu.** Solutions of nonlinear oscillators with fractional powers by an iteration procedure, *J. Sound Vib.* 294 (2006) 608.
- [37] **H. Hu.** Solutions of a quadratic nonlinear oscillator: iteration procedure, *J. Sound Vib.* 298 (2006) 1159.
- [38] **S. K. Lai, C. W. Lim, B. S. Wu, C. Wang, Q. C. Zeng, X. F. He.** Newton–harmonic balancing approach for accurate solutions to nonlinear cubic–quintic Duffing oscillators, *Applied Mathematics Modeling*, (2008) In press.
- [39] **J. I. Ramos.** On Lindstedt–Poincaré techniques for the quintic Duffing equation, *Applied Mathematics and Computation* 193 (2007) 303.
- [40] **R. R. Puseňjak.** Extended Lindstedt–Poincaré method for non-stationary resonances of dynamical systems with cubic nonlinearities, *Journal of Sound and Vibration* 314 (2008) 194.
- [41] **H. Hu, Z. G. Xiong.** Comparison of two Lindstedt–Poincaré-type perturbation methods, *Journal of Sound and Vibration* 278 (2004) 437.
- [42] **P. Amore, A. Aranda.** Improved Lindstedt–Poincaré method for the solution of nonlinear problems, *Journal of Sound and Vibration*, 283 (2005) 1115.
- [43] **S. S. Ganji, A. Barari, D. D. Ganji.** Approximate analyses of two mass-spring systems and buckling of a column, *Computers & Mathematics with Applications*, 2011, 61(4), 1088-1095.
- [44] **M. Momeni, N. Jamshidi, A. Barari, G. Domairry.** 2010, Application of He's Energy Balance Method to Duffing Harmonic Oscillators, *International Journal of Computer Mathematics*, 88(1), 135-144, DOI:10.1080/00207160903337239.
- [45] **T. Özis, A. Yildirim.** Determination of periodic solution for a $u^{1/3}$ force by He's modified Lindstedt–Poincaré method, *Journal of Sound and Vibration*, 301 (2007) 415.
- [46] **W. P. Sun, B. S. Wu,** Large amplitude free vibrations of a mass grounded by linear and nonlinear springs in series, *Journal of Sound and Vibration* 314 (2008) 474.
- [47] **M. Omidvar, A. Barari, M. Momeni, D. D. Ganji.** 2010, New class of solutions for water infiltration problems in unsaturated soils, *International Journal of Geomechanics and Geoenvironment*, 5(2), 127 – 135.
- [48] **S. K. Lai, C. W. Lim.** Accurate approximate analytical solutions for nonlinear free vibration of systems with serial linear and nonlinear stiffness, *Journal of Sound and Vibration* 307 (2007) 720.

- [49] **J. H. He.** Some asymptotic methods for strongly nonlinear equations, *International Journal of Modern Physics B*, 10 (2006) 1141
- [50] **B. Liu, Z. Q. Yue, L. G. Tham.** Analytical design method for a truss-bolt system for reinforcement of fractured coal mine roofs—illustrated with a case study, *International Journal of Rock Mechanics & Mining Sciences* 42 (2005) 195.
- [51] **Seongkyu Lee, Kenneth S. Brentner, F. Farassat, Philip J. Morris.** Analytic formulation and numerical implementation of an acoustic pressure gradient prediction, *Journal of Sound and Vibration* 319 (2009) 1200.
- [52] **A.V. Metrikine.** Steady state response of an infinite string on a non-linear visco-elastic foundation to moving point loads, *Journal of Sound and Vibration* 272 (2004) 1033.
- [53] **J. I. Ramos.** Linearized Galerkin and artificial parameter techniques for the determination of periodic solutions of nonlinear oscillators, *Applied Mathematics and Computation* 196 (2008) 483.
- [54] **J. I. Ramos.** An artificial parameter–Linstedt–Poincaré method for oscillators with smooth odd nonlinearities, *Chaos, Solitons and Fractals* , doi:10.1016/j.chaos.2008.01.009 .
- [55] **J. I. Ramos.** An artificial parameter Linstedt–Poincaré method for the periodic solutions of nonlinear oscillators in which the restoring force is inversely proportional to the dependent variable, *Journal of Sound and Vibration* 318 (2008) 1281.