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Considerations on the H_2 desorption process for a combination reactor based on metal and complex hydrides

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Abstract

Hydrogen storage systems based on the combination reactor concept are promising for application of future complex hydride materials with high storage capacities and low reaction kinetics at moderate operating temperatures. In such a reactor, a fast reacting metal hydride is added to a complex hydride material in a separate compartment of the tank combining the advantages of the high storage capacity of the complex hydride with the high reaction rate of the metal hydride. In the present publication, three issues regarding the desorption performance of such a reactor are discussed based on analytical considerations and 1D simulations. First, it is studied whether the optimal reactor design based on a tubular geometry that has been previously determined for the absorption reaction also enables satisfying desorption performances. It can be concluded from the corresponding simulations that based on the properties of the present reference materials LaNi_{4.3}Al_{0.4}Mn_{0.7} and 2 LiNH₂-1.1 $MgH_2-0.1LiBH_4-3$ wt.% ZrCoH₃ the hydrogen desorption performance of the materials in this reaction geometry is good. Second, it is shown that besides the geometry of the reactor, also its module size is important, as it can be crucial for the thermal management during the desorption. A methodology was developed that allows to analytically determine a first estimate for the best minimum module size configuration - only based on the desorption rate of the basic material. This approach is confirmed by time dependent 1D simulations applying a validated model for the reference materials. Third, the influence of a realistic periodic desorption load on the performance of a combination reactor is studied. The results clearly show that since the addition of a MeH material enables much smaller module sizes, it is advantageous for the thermal management of complex hydride based reactors and increases their flexibility.

Keywords

Complex hydride, metal hydride, combination tubular reactor, modelling, desorption

1. Introduction

It is discussed in several scenarios of our future energy system that hydrogen can play an important role as environmental friendly energy carrier [1,2]. However, one bottleneck for a closed hydrogen cycle is still the efficient storage of this lightweight gas. Amongst others, one possibility is the utilization of solid state storage materials, which is a rather safe and long-term storage option. However, so far suitable materials still lead to quite heavy systems [3]. Thus, recently the focus of the application of these materials has been shifted from passenger cars to heavy duty vehicles, e.g. forklifts [4,5]. Besides classical metal hydrides (MeH) that are already considered for real-life appliations, promising solutions based on complex hydrides (CxH) are still under investigation. These kind of materials show

improved H₂ storage capacities, thus they are in principle able to overcome the obstacle of poor gravimetric storage density common for metal hydrides [6,7]. However, these materials usually show low reaction rates at temperatures below 100 °C due to kinetic limitations of their conversion reaction with hydrogen [8,9]. One possibility to use the high storage capacities of these materials and to overcome their limitation in reaction dynamics is a combination reactor concept that has been presented by the authors before [10,11]. The idea of this concept is the combination of a fast reacting metal hydride material with a slowly reacting complex hydride material that shows a very high storage capacity. In these previous works, this concept has been presented and discussed including a model validation for the absorption and the desorption processes. Based on the validated model for two reference materials, it is now possible to investigate various suitable tank designs. E.g., in a recent study, the absorption process has been investigated and an optimal geometry for the reference materials has been defined [12]. As for motive applications the fuelling of the gas has to be very fast, the absorption process is very sensitive towards reactor geometry and material properties. Thus, it is likely that the absorption requirements will define the overall geometry of a corresponding tank. However, for technical applications, not only the absorption process, but also the desorption process has to be considered. The most common scenario for a desorption process in mobile applications is the coupling to a HT-PEM fuel cell, where the exhaust heat of the fuel cell is used in order to supply the required heat for the desorption reaction to the hydride storage material [13-15]. For complex hydrides the desorption rates at fuel cell relevant pressures > 1.7 bar and temperatures (HT-PEM, 170 °C) are usually quite low. Therefore, it is not trivial to combine high storage efficiencies (full capacity) with fast discharging dynamics in a practical application, e.g. due to heat management problems.

In the present paper, the main characteristics of a desorption process as well as the resulting challenges are discussed considering the two reference materials that have previously been used for model validation: LaNi_{4.3}Al_{0.4}Mn_{0.7}, in the following abbreviated as MeH material, and 2 LiNH₂– $1.1MgH_2$ – $0.1LiBH_4$ –3wt.% ZrCoH₃, in the following abbreviated as CxH. It is obvious that with the effective storage capacities of 1.2 wt.% and 3.2 wt.% for the MeH and CxH material, a final reactor design will still not be sufficient to fulfil the system target values of the DOE for gravimetric and volumetric densities. However, for these two materials, validated models exist and the general principle of the combination reactor concept has been proven. Thus, these materials are chosen as reference materials for the present publication.

The design of the basic reactor geometry for this study has been taken from the previously published optimization process of the absorption reaction [12]. Starting from this publication, it will be first checked whether this geometry is also suitable for a generalized desorption scenario. Then in the second part, a new aspect – the module size of a modular hydride reactor – is discussed, since it influences the thermal management of the reactor and thus highly affects the desorption process (e.g., coupling to a fuel cell. In order to clarify the wording in this manuscript, the *size of a module* always refers to the *mass of hydride* that desorbs hydrogen *at the same time*. Furthermore, when the desorption rate is mentioned it corresponds to the effective desorption reaction rate of the pure material, while the discharging rate corresponds to the discharging process of the overall tank.

Finally, in the third part, more realistic dynamic operation scenarios are discussed. Therefore, the basic model is extended to capture the MeH reabsorption process of hydrogen during low load conditions.

1 Model formulation

1.1 Geometry

For modelling purposes, in this paper a simple axisymmetric 1D geometry has been applied that represents the cylindrical shape of a tube-shaped reactor, compare Figure 1 and [12]. The metal hydride material is located in the center with a radius $r_{MeH} = d_{MeH}$. In the annulus of the tube, the complex hydride is inserted and the thickness of this layer can be varied corresponding to the ratio $\xi = d_{CxH}/d_{MeH}$. The resulting volumetric fraction of the CxH material is referred to as $\varepsilon_{CxH} = V_{CxH} / V_{total}$. Both materials are separated by a gas permeable separation layer (GPSL). As this layer consists of a thin stainless steel mesh, the heat and mass transfer resistances at this boundary are negligible. The thickness of the stainless steel wall $d_{S,S}$ separating the hydride container from the heat transfer fluid is considered according to the following equation [12]

$$d_{\rm S.S} = \frac{2r_o}{1 + \frac{20}{S}\frac{K}{P}v},$$
(1)

where r_0 is the external diameter of the stainless steel tube and *P* is the maximum applied pressure. The remaining parameters have been taken for stainless steel 1.4571 at a maximum temperature of 250 °C: design strength *K* = 186 Nmm⁻², safety factor *S* = 1.5 and the utilization factor of the allowable design stress *v* = 0.065. The geometries used for simulations in this study are summarized in Table 1.

1.1.1 Model equations

For the presented simulations, the model is implemented into the FEM software COMSOL Multiphysics and rate equations, the mass balance of the gas as well as the total energy balance are considered for the metal hydride and the complex hydride domains. For the steel wall, only an energy balance of the solid is considered. The values for the respective properties as well as all assumptions can be taken from the previous publication, where this model has been validated [11].

The rate equations can be written as

$$\frac{\partial X}{\partial t} = r_{\rm R}.\tag{2}$$

where X is the transformed fraction, $r_{\rm R}$ is the effective reaction rate for the materials as function of temperature and pressure given in s⁻¹. The corresponding equations are summarized in the following Equations 3-5. More details can be found in [10,11,16].

Desorption of the CxH material (Li-Mg-N-H), 1^{st} step, $0 < X_1 < 0.67$:

$$r_{\rm R,des,CxH,I} = \frac{\partial X_{\rm I}}{\partial t} = 2.35 \cdot 10^{12} \, [\rm s^{-1}] \cdot \exp\left(\frac{-131800[\rm Jmol^{-1}K^{-1}]}{\Re T}\right) \cdot 1.5(1 - X_{\rm I})[-\ln(1 - X_{\rm I})]^{0.33} \cdot \ln\left(\frac{P_{\rm eq}}{P}\right) \tag{3a}$$

Desorption of the CxH material (Li-Mg-N-H), 2^{nd} step, 0.67< X_{II} <1:

$$r_{\rm R,des,CxH,II} = \frac{\partial X_{\rm II}}{\partial t} = 3.044 \cdot 10^{15} \, [\rm s^{-1}] \cdot \exp\left(\frac{-161400[\rm Jmol^{-1}K^{-1}]}{\Re T}\right) \cdot \left(1 - \frac{0.001515}{wt_{\rm max} \cdot 0.33}(P - 1.1[\rm bar])\right) \tag{3b}$$

Desorption of the MeH material:

$$r_{\mathrm{R,des,MeH}} = \frac{\partial X}{\partial t} = 20 \, [\mathrm{s}^{-1}] \cdot \exp\left(\frac{-16500[\mathrm{Jmol}^{-1}\mathrm{K}^{-1}]}{\mathscr{H}T}\right) \cdot (1 - X) \cdot \ln\left(\frac{P_{\mathrm{eq}}}{P}\right). \tag{4}$$

Absorption of the MeH material:

$$_{\text{R,abs,MeH}} = \frac{\partial X}{\partial t} = 100 \, [\text{s}^{-1}] \cdot \exp\left(\frac{-21000[\text{Jmol}^{-1}\text{K}^{-1}]}{\Re T}\right) \cdot (1 - X) \cdot \left(\frac{P - P_{\text{eq}}}{P_{\text{eq}}}\right).$$
(5)

In case of "reabsorption" conditions during the desorption reaction (see Section 2.3), $r_{\rm R}$ is described by an IF condition: IF the system pressure is below the desorption equilibrium pressure, the equations for $r_{\rm R,des,MeH}$ are valid. IF the system is above the absorption equilibrium pressure and IF already more than 1% of the transformed fraction has been desorbed, $r_{\rm R}$ is characterized by the equations for $r_{\rm R,abs,MeH}$.

When Darcy's law, $\nabla P_g = -\frac{\mu}{\kappa} \cdot \vec{v_{g'}}$ is applied for the description of the gas velocity, $\vec{v_g}$, the equation of the mass balance of the gas phase in terms of gas pressure P_g is given by

$$\varepsilon \frac{M}{\Re T} \frac{\partial P_{\rm g}}{\partial t} - \varepsilon \frac{P_{\rm g} M}{\Re T^2} \frac{\partial T}{\partial t} - \nabla \left(\rho_{\rm g} \frac{K}{\mu} \nabla P_{\rm g} \right) = -(1-\varepsilon) r_{\rm R} w t_{\rm max} \rho_s - \dot{m}_{des},\tag{6}$$

where *M* is the molar mass of H₂, \Re is the real gas constant, *T* the temperature, ρ_g the density of the gas, *K* the permeability of the powder bed, μ the viscosity of H₂, ε the porosity of the bed, *wt*_{max} the maximum hydrogen storage capacity of the material and ρ_s the solid density. \dot{m}_{des} refers to the mass of hydrogen that is removed from the reactor, see Section 1.1.2.

For the *energy balance of the system*, local thermal equilibrium (LTE), $T = T_g = T_s$, is assumed leading to

$$[(1-\varepsilon)c_{\mathrm{p},\mathrm{s}}\rho_{\mathrm{s}} + \varepsilon c_{\mathrm{p},\mathrm{g}}\rho_{g}]\frac{\partial T}{\partial t} = c_{\mathrm{p},\mathrm{g}}\rho_{\mathrm{g}}\frac{K}{\mu}\nabla P_{\mathrm{g}}\nabla T - \nabla \cdot (-\lambda_{\mathrm{eff}}\nabla T) -(1-\varepsilon)r_{\mathrm{R}}wt_{\mathrm{max}}\rho_{s}\Delta_{\mathrm{R}}H.$$
(7)

The expression on the left hand side of Equation 7 refers to the accumulation of enthalpy in the gas as well as in the solid phase with the heat capacity of the gas and solid $c_{p,s/g}$. On the right hand side, convective heat transfer of the gas phase, heat transfer by thermal conduction in the powder and a heat source due to the reaction are accounted for, where $\Delta_R H$ is the enthalpy of the desorption reaction. In case of "reabsorption", in analogy to the IF condition for the effective reaction rate mentioned above, another IF condition has been applied on the enthalpy of reaction.

1.1.2 Initial and boundary conditions

The initial condition for the temperature of the tank is given by the constraints of a HT-PEM fuel cell, $T_i = T_{\text{steel},i} = 170 \text{ °C}$, and the initial pressure is set to $P_i = 20$ bar. This pressure is below the fuelling pressure of 70 bar, however, as in the first seconds H₂ is only released from the void space of the reactor, this behavior is not relevant for the present considerations. Furthermore, in the initial state, both materials are fully in the absorbed state.

For the energy equation, the boundary condition at the center is a symmetry condition. At the outer surface, a heat flux from the steel wall to the heat transfer fluid has been assumed that is characterized by a heat transfer coefficient of $\alpha = 600 \text{ Wm}^{-2}\text{K}^{-1}$, compare [11]. The most important boundary condition during desorption experiments, however, is given by the mass flow rate of H₂ that

is removed from the system and consumed by the fuel cell. This flow rate is implemented as a (negative) source term in the gas balance of the system (see Equation (6)), and it causes the characteristic decrease in system pressure that results in the endothermal desorption reaction and the corresponding temperature decrease. Thus, this mass flow rate is a very important variable for the simulations and has to be chosen in a reasonable way. For the present simulations, the storage capacity of the reactor has been related to the storage discharging time in order to be able to compare different scenarios and different reactor designs. E.g., in the present basic case it has been assumed that the storage capacity of the MeH and the CxH in a reactor with a length of 1 m is deploited in 3 h. Thus, the required desorption H₂ flow rate r_{des} is given in s⁻¹ as 1/10800 s = 9.26 10⁻⁵ s⁻¹, and the resulting mass flow rate is calculated as

$$\dot{m}_{\rm des} = \rho_{\rm s} w t_{\rm max} (1 - \varepsilon) \cdot r_{\rm des} \cdot A_{\rm tube} \cdot H, \tag{8}$$

where A_{tube} refers to the perpendicular cut of tube axis that corresponds to the geometry of the model and the height of the reactor is defined as H = 1 m. For the termination of the simulation, the following condition has been applied: as soon as the pressure reaches a value below 1.7 bar - the minimum working pressure to supply the fuel cell - the simulation is terminated. Besides simulations with a constant desorption H₂ flow rate, also simulations with periodic H₂ flow rates (thus, varying values of the source term) have been performed, see Section 2.3.

2 Results and Discussion

The general effect of the addition of a MeH material to a CxH material in a combination reactor has already been discussed in a previous publication [11] and can be summarized as follows:

- The addition of the MeH to the CxH material stabilizes the pressure in the system: As the reation rate of the MeH is several orders of magnitude higher than the rate of the CxH, the pressure in the system will not fall below the equilibrium pressure of the MeH material as long as the MeH can desorb hydrogen.
- The efficiency of the hydrogen storage in the CxH material is improved since, as long as the MeH is stabilizing the pressure, the CxH material can desorb at its own desorption rate. Thus, the MeH is just desorbing as much H₂ as to meet the demand of hydrogen for the fuel cell.
- The temperatures in the reactor decrease due to the endothermic reaction. As the MeH is
 placed at the center of the tube, where the lowest temperatures appear, this region of the
 reactor is efficiently used as the MeH still shows sufficient kinetics even at low temperatures.
 (< 150 °C).

The goal of the present publication is to study these effects in more detail and derive suggestions for future tank design of similar reactors. First, the effect of varying geometries is studied in Section 2.1, and the optimum reactor design for absorption determined in a previous publication is evaluated with respect to desorption constraints. Then, in Section 2.2, the minimum module size for a satisfying desorption performance is discussed, when the effective reaction rates of the materials are known. This minimum module size is important for applications requiring a substantial thermal management, e.g. sufficient heat transfer at a specific temperature level, when a tank is coupled to a HT-PEM fuel cell. Finally, the dynamic performance of a combination reactor is studied when the H_2 flow rate shows

a dynamic behavior, and the observed effects are explained by the effective desorption rates of the different materials, see Section 2.3.

2.1 Optimum reactor design for desorption

In a previously published study, an optimum geometry for a combination reactor has been determined with respect to a short filling time leading to a reactor with a geometry of $d_{MeH} = 10$ mm and $\xi = 1.25$ [12]. In this section, it is now studied whether this radial geometry determined for absorption is also useful for the constraints that apply during desorption operation. Therefore, starting from the optimum absorption geometry, different geometric parameters are varied in the following, where always a scenario with a complete desorption in 3 h is considered.

2.1.1 Variation of the ratio ξ

First of all, the effect of varied values for ξ on the desorption performance is studied in analogy to the absorption study [12]. Figure 2 shows the corresponding transformed fraction plot for ξ values of 0.5 to 4, for a discharging time of 3 h. Additionally, the values for volumetric ratio ($\varepsilon_{CxH} = V_{CxH}/V_{total}$), storage capacity and stored H₂ per m length of reactor are summarized in Table 1 for the different cases (A, D, E, F). In case of absorption there exists a clear optimum regarding the minium fuelling time as it has been shown in a previous publication [12]. In contrast, for the desorption process the decision on an optimum is not clearly defined as there is not only one single criterium that has to be met. For the combination reactor, the optimal value is rather a trade-off between discharging efficiency and CxH volumetric ratio. For a value of $\xi = 0.5$ (green), the discharging efficiency is very high, however, with $\varepsilon_{CxH} = 0.56$, the tank mainly consists of MeH material. On the other hand, for $\xi = 4$ (turquoise), the discharging efficiency is very low as the desorption is interrupted early. Therefore, a value of 1.25 (black), which has been the optimum for absorption is also suitable for the discharging process leading to a CxH fraction of $\varepsilon_{CxH} = 0.8$.

2.1.2 Variation in d_{MeH}

Next to a variation of ξ , also the radius of the MeH material d_{MeH} at the center has been varied from 10 to 5 and 15 mm, and the optimum values for each ξ have been adapted from the analogous absorption simulations. Figure 3 shows the corresponding absolute mass of hydrogen that is desorbed in each case per m length of reactor (compare also Table 1). From this plot it is obvious that in case of the reactor with d_{MeH} =15 mm (red), the MeH dominates the combination reactor as more hydrogen is actually stored in the MeH (23.5 g, dashes) than in the CxH (16.5 g, solid). In case of the 5 mm reactor (orange) this ratio is much better with 2.6 to 8.2 g. However, the total mass of H₂ stored per m length of reactor is very small. Thus, it can be concluded that for desorption a geometry using $d_{\text{MeH}} = 10$ mm (black), shows a good trade off between performance and stored H₂ per m.

Summarizing the parameter study on the optimum reactor geometry, it is obvious that the optimum geometry for absorption is also applicable for the desorption process.

2.2 Discussion of minimum module size for desorption process

As it has been concluded in the previous section, a reactor design that is optimal with regards to the absorption process - filling time - is also suitable for a general desorption scenario. However, for the desorption process also the module size of the overall tank can be crucial, as it will be shown in the following.

In this section, the correlation between the effective reaction rates of the materials as well as the discharging time of the storage reactor and the minimum possible reactor module size is discussed. Furthermore, it is introduced how this module size can influence the thermal management of a reactor during desorption operation. As mentioned before, the module size always refers to the mass of hydride that desorbs hydrogen at the same time, thus it is independent of any geometric assumption.

First, two basic scenarios are depicted in Figure 4, in order to explain the effects of different module sizes on the operation procedure. In both cases, a total mass of 5 kg_{H2} is stored in 10 elements of the same dimensions and the reactor is operated for 10 h until it is compeletely discharged:

- I. In the 1st case, 5 (out of 10) elements are connected to one module. They are heated up to the nominal operation temperature and operated for the first 5 h. Each element is desorbing hydrogen with a rate of 0.1 kg_{H2} per h resulting in an overall H₂ discharging rate of 0.5 kg_{H2}h⁻¹ of the whole module. After 5 h, the second module has to be in operation. Therefore, the next 5 identical elements are heated up and the same H₂ flow rate is desorbed by the material in this module. So, hydrogen has been delivered at a discharging rate of 0.5 kg_{H2} h⁻¹ to the fuel cell during 10 h of operation. In this case, it is possible to use hydride materials with very low effective (desorption) reaction rates, as the size of the two modules is rather big, and the material in the single elements has to desorb H₂ at a rather low rate. However, this system is not very flexible concerning cold start ability and/or heat losses, as a very big part of the overall tank has to be heated up and kept at operation temperature at the same time.
- II. In the 2nd case, only 1 element is heated up to operation temperature at the same time, so in this case one element refers to one module. Thus, in this module the effective desorption reaction rate of the materials has to be sufficiently high to discharge 0.5 kg_{H2} h⁻¹ for one hour. Then, the next module is heated up and desorbs the same amount of H₂ and so on. Therefore, also in this case a constant H₂ discharging rate of 0.5 kg_{H2} h⁻¹ can be realized while only 1 module is kept at operation temperature at the same time. This kind of configuration is advantageous when a more flexible operation behavior is desired, especially considering the start-up of the system.

From these two cases it is clear that it is desirable to use the smallest possible module size for an enhanced thermal management allowing for a flexible operation of the tank. In the following section, it will be discussed how the minimum module size for a H₂ storage reactor can be determined analytically by the effective desorption reaction rate of the material. The considered boundary conditions for these analytical considerations are given by a HT-PEM fuel cell with an assumed temperature of the heat transfer fluid $T_{\text{HTF}} = 165 - 170 \,^{\circ}\text{C}$ and a minimum H₂ supply pressure of 0.5 - 1 bar and for the material properties of the two reference materials.

2.2.1 Analytical Considerations

For a first guess on the minimum modules size, it is sufficient to consider only the effective reaction rates of the materials. The corresponding procedure can be seen in a plot showing the reaction rate versus the transformed fraction for the considered temperatures and pressures (170 °C and 0.5 – 1 bar). For the reference materials considered in this study, this kind of plot is shown in Figure 5 A and B. First of all, from this plot it is clear that the reaction rates of the MeH material (B) are 2 to 3 orders of magnitude higher than the reation rates of the considered CxH material (A), see different scales. The straight lines indicate the required rates for a complete desorption in $t_{des} = 2$, 3 or 5 h. As the effective rate of the MeH is higher than these rates even up to a fully discharged state, it is obvious that the MeH material can continuously desorb H₂ at a high rate. In contrast, the effective reaction rates for the CxH material are much lower. In this case the material requires at least 5 h for a complete discharging process, as only in this case the reaction rate is higher than the theoretical rate for a full conversion in 5 h (a rate of $1/(5 \cdot 3600 \text{ s}) = 5.56 \cdot 10^{-5} \text{ s}^{-1}$). Therefore, for this material, a desorption time of 2 or 3 hours will only lead to a H₂ release from the 1st desorption step.

Thus, it can be concluded that for the MeH material, there does not exist a minimum module size from the point of view of reaction rates, as even the total capacity can be released in few seconds. However, for the CxH material, the size of the modules should be chosen in a way that each module can desorb H_2 for at least 5 h in order to be able to utilize the total capacity of the material. So, comparing Case I and II in Section 2.2.1, each module has to consist of several elements.

Besides the effective reaction rates of the materials at isothermal and isobaric conditions, in a second step it is useful to include more information on the actual reactor design (tube diameters) in the considerations for the minimum module size. As the reactor design will influence the temperature profile appearing in the reactor, this will also influence the overall reactor performance and the assumption of isothermal conditions. E.g., for larger tube diameters the temperature at the center will decrease due to the endothermic reaction, thus, the temperature of the heat transfer fluid will not represent the temperature of the material, consequently the assumption of constant *T* and *P* is not valid any more. For the case of an annular ring filled with CxH material, the temperature decrease between the heat transfer fluid and the temperature T_2 at the GPSL, can be calculated analytically according to the following equation [17], assuming a constant heat flux due to the endothermal desorption reaction of a constant H₂ flow rate from the CxH material,

$$T_{\rm HTF} - T_2 = -\frac{(\Delta_{\rm R} H\rho)_{\rm CxH}(1-\varepsilon)}{4\lambda_{\rm eff} t_{\rm dis}} \left(r_{\rm i}^2 - d_{\rm MeH}^2\right) + \frac{(\Delta_{\rm R} H\rho)_{\rm CxH}(1-\varepsilon)}{2\lambda_{\rm eff} t_{\rm dis}} d_{\rm MeH}^2 \ln\left(\frac{r_{\rm i}}{d_{\rm MeH}}\right),\tag{9}$$

where T_{HTF} and T_2 are the temperatures of the heat transfer fluid and the GPSL, $\Delta_{\text{R}}H$, ρ_{s} , wt_{CxH} , ε and λ_{eff} are the reaction enthalpy, the density, the storage capacity, the porosity and the effective thermal conductivity, and r_{HTF} and d_{MeH} are the radii of the annulus corresponding to Figure 1. Furthermore, t_{dis} is the given discharging time. The calculated temperature decreases for the optimum design determined in the absorption scenario [12] (compare Table 1, A, B, C), are 7.1 K, 4.7 K and 3.5 K for the complete desorption in 2 h, 3 h and 5 h, respectively.

Thus, for the 3 h and 5 h desorption the temperature decrease is not very prominent (< 5 K) and it can be assumed that the first estimation of the minimum module size using only the effective reaction rate information at constant temperature is correct.

2.2.2 Simulation results

The previously introduced analytical considerations showed the challenges during desorption processes for materials with very low effective desorption reaction rates, e.g. CxH materials. Furthermore, they showed how a first estimate on the minimum possible module size for a combination reactor can be obtained, based on the assumption that the CxH material is dominating the overall performance. In this section, this initial analytical estimate is analyzed in detail by a discussion of the time dependent desorption simulations of both materials in the combination reactor as well as by a discussion of the final mass of H_2 that could be desorbed in the different cases.

Figure 6A shows the transformed fraction of MeH and CxH versus time profiles for the simulated cases, where the total stored H₂ capacity is desorbed in 2 h, 3 h and 5 h (compare Table 1, A, B, C). Regarding the geometry, the optimum geometry obtained from the 1D simulations for absorption is considered with $d_{CxH} = 10$ mm and $\xi = 1.25$, and the length of one module is defined as 1 m.

The time dependent behavior is in all three cases quite similar and can be described as follows: in the beginning the straight lines which indicate the mass H₂ desorbed by the CxH material increase while the dashed lines referring to the MeH material stay at a very low level. Thus, the majority of H₂ is desorbed by the CxH material. Then, towards the end of each scenario, the effective reaction rate of the CxH is slowing down and the MeH material starts to desorb H₂ filling the gap to the required H₂ flow rate for the fuel cell. The same effect can be seen from Figure 6B where the temperature at the GPSL, *T*₂ (dashes), as well as the temperature at the center, *T*₁ (solid), are plotted versus time for the same three cases. First, the temperature in the center follows the equilibrium of the CxH indicating that mainly CxH material is desorbing hydrogen. Then, more H₂ has to be desorbed by the MeH material, and the temperature *T*₁ in the center starts to vary significantly from the temperature at the GPSL, *T*₂.

This behavior of the three different desorption cases is in accordance with Figure 5, that has been discussed above: in all three cases, towards the termination of the simulation the MeH is compeletely discharged with a wt_{frac} of 0.009 at 165 °C (dashed lines), as the effective reaction rate of this material is very high even at the applied pressures and temperature. However, for the discharge of the CxH material different values are obtained: in case of a discharging within 2 h and 3 h, just 2 wt.% and 2.4 wt.% can be desorbed corresponding mainly to the 1st desorption step. Only for the case of a desorption for 5 h the full H₂ capacity of 3.2 wt.% can be reached. Linking these findings to the general exemplary cases in Section 2.2.1, it is obvious that for a discharging of the reaction in 2 or 3 h, the currently used module size is too small to completely discharge the reactor for the effective reaction rate of the reference materials. Therefore, if a complete discharging is desired, either the module size should be increased or the discharging rate reduced (to e.g. 5 h).

Concluding this section it can be stated that the analytical considerations and the simulated results agree very well: The minimum discharging rate from 3 h for the complete 1st desorption step and 5 h for a full 2nd step desorption that have been predicted by the analytical considerations have been

confirmed in detail by the 1D simulations. For a discharging process within 2 h, significantly less H_2 could be desorbed. Here a clear deviation from the analytical estimation is obtained since it predicted a very similar behavior to the discharging within 3 h. However, in this case, the more prominent temperature decrease - that has already been predicted analytically - deteriorates the performance, as it can be clearly seen from Figure 6B.

Thus, the analytical considerations presented in Section 2.1.1 can be useful to understand the correlations between the module size of a reactor and its desorption performance at a given discharging rate for materials with given desorption rates. This fact can especially be interesting for tank applications requiring a substantial thermal management, as it is the case for coupling a solid state hydrogen storage tank to a HT-PEM fuel cell.

2.3 Simulations with varying loads

In the previous section, it has been introduced why small module sizes can be an important issue for technically relevant storage tanks. This was done based on the assumption of constant discharging rates. In this section, more realistic boundary conditions, for e.g. a scenario where the tank is coupled to a HT-PEM fuel cell, will beconsidered. Therefore, besides scenarios with constant discharging rates simulating a constant power output of the fuel cell, the behavior of a tank during varying loads is of high practical importance, e.g., in a driving scenario this can correspond to going uphill or going downhill [18].

In order to understand the performance of a combination reactor during dynamic loads and especially the effect of the additional MeH, 3 different scenarios are simulated using the basic geometry with d_{MeH} = 10 mm and ξ = 1.25 (see Table 1, A):

- The first scenario (I) is the basic case for the optimum geometry with a discharging time of 3 h, as it has been used also in the previous sections.
- The second scenario (II) refers to the same discharging time, however, in this case the rate is not constant but it is varied: for 30 s, 6 times the base load is discharged and for the following 450 s, the load is reduced by a factor of 10 to 0.67 times the base load.
- The third scenario (III) refers again to the same discharging time, but now the rate is varied between 30 s and 450 s from 0.53 to 8, so by a factor of 15.

Figure 7 shows the discharging rates for the three different cases together with the rates versus transformed fraction plot of the CxH material at given temperature and pressure (T_{HTF} = 170 °C and P > 1.7 bar, compare Figure 5A). From this plot it becomes obvious that using the constant discharging rate of Scenario I (blue straight line), the 1st desorption step of the CxH material can in principle be fully discharged as the required flow rate is below the possible desorption flow rate up to 60% transformed fraction. In Scenario II, the peak rates should be acceptable up to a transformed fraction of approx. 30%, thus in this case approx. half of the 1st desorption step should be available for this varying load case. Finally, for Scenario III, the peak loads exceed the possible desorption rates of the CxH material even at very low transformed fractions. Thus, in a pure CxH reactor with the present geometry and module size, the material would not be able to keep up the required discharging flow rate from the fuel cell even from the beginning – even for ideal (isothermal and isobaric) conditions.

For scenarios with varying discharging rates, the analytical considerations reflecting mainly the CxH material desorption rates are insufficient to capture the final behavior. Therefore, detailed simulations are required that take the simultanious reactions of both materials into account. Figure 8 shows the results of the time dependent simulations of the three scenarios using the optimum geometry of the combination reactor (see Table 1, A). In Figure 8A, the system pressure is plotted versus time for the three scenarios. Obviously, in Scenarios II and III, the pressure is varying periodically due to the periodic discharging rate. The intensity of this fluctuation depends on the intensity of the different discharging rates, thus for Scenario III (green, dots) the variations are more significant than for Scenario II (red, dashes). Furthermore, it is obvious that in all three scenarios the time until the experiment is interrupted is very similar. This is surprising as the rate during peak loads in Scenario III clearly exceeds the possible effective desorption rate of the CxH material, see discussion above. This behavior is due to the additional MeH material in the combination reactor and it clearly shows the advantages on the dynamic behavior of such a combination reactor: even when in a scenario the peak loads exceed the maximum desorption rates of the CxH material, the MeH material can additionally desorb and thus stabilize the system pressure. Furthermore, as the system pressure is stabilized, it is possible that in all three cases the same amount of H₂ is desorbed from both materials as it is shown in Figure 8B, where the mass of desorbed H_2 is plotted: in all three cases not only the MeH but also the CxH material desorbs H_2 up to the same value.

In the following, the two parallel reactions are discussed in more detail. It will be shown that the MeH material is not only desorbing extra hydrogen during peak loads, but it is also able to "reabsorb" hydrogen during low loads. Thus, the "buffering" function of the MeH material is not only available once, but it can be recharged. Figure 9 shows a detail of the system pressure (red) as well as the equilibrium pressures for desorption of the CxH (black dots) and the MeH (grey dots) and for absorption for the MeH (grey dashes) for Scenario III. From this graph it is obvious that the system pressure never falls below the desorption pressure of the MeH material as this is desorbing sufficiently fast to keep a constant system pressure. Furthermore, during low load conditions the system pressure never exceeds the absorption equilibrium pressure of the MeH material indicating that the material "reabsorbs" hydrogen. Since the system pressure is constantly below the desorption equilibrium pressure of the CxH material, the CxH desorbs continuously thereby also supplying hydrogen for the "reabsorption" in the MeH. The "reabsorption" is possible until approx. 3600 s, then the system pressure does not reach the required absorption pressure of the MeH anymore. The same fact is shown in Figure 10, where a detail of the desorbed mass H₂ is plotted versus time only for the MeH material. Here, it can be clearly seen that the desorbed mass of H_2 in the MeH material is decreasing during low loads and - during the first part - increasing during peak loads.

In summary it can be concluded that especially Scenarios II and III clearly show the advantages of the combination reactor concept during a dynamic desorption scenario: even when the required peak discharging rate exceeds the possible effective desorption rate of the CxH material, it is possible to show a satisfying performance on a reactor level. Thus, as long as the required discharging hydrogen rate can be on average supplied by the CxH material in a given geometry and module size, even much higher peak discharging rates can be tolerated in a combination reactor as the MeH material can desorb and reabsorb sufficiently fast.

Coming back to the discussion on the minimum module size in Section 2.1, it is obvious that the addition of some MeH to the CxH material can decrease the required module size for a pure CxH tank and thus increase the flexibility of the tank. Especially for the start-up phase or during mode changes in the operation, that require a substantial thermal management, a reactor with a suitable combination of materials can improve the overall performance.

3 Conclusions

In the present publication, the desorption performance of different configurations of a combination reactor based on a reference metal hydride and a reference complex hydride have been studied using analytical methods and simulations of a 1D model using the software COMSOL.

First, a sensitivity study of the desorption performance has been performed by varying the ratio between the MeH and the CxH as well as the radius of the inner MeH tube starting from the previously determined optimal absorption geometry. Both studies indicated, that the optimal absorption geometry is also suitable for the given desorption conditions.

Second, based on generic operation conditions, the influence of the module size of a reactor on the required thermal management during operation was analysed. Then, the minimal module size of a reactor has been discussed when a certain discharging time is required and the effective desorption reaction rates characterizing the CxH material are known. As the rates of the present reference material at 165 °C and 1 bar are quite low, very large modules are required in order to be able to deliver sufficient hydrogen. Thus, for these materials, the thermal management of the overall tank will be challenging.

Finally, the desorption process has also been studied for varying loads, e.g. for a load variation by a factor of 10 and 15. In this case it was shown that the combination reactor is advantageous as the MeH material was able to buffer and reabsorb hydrogen during peak and low loads, respectively, leading to smaller possible module sizes than analytically expected. Therefore, based on the combination principle the desorption performance on a reactor level can be better than indicated by the effective desorption rates of the respective CxH material.

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Figures

Figure 1: Schema of the modelled 1D geometry of an axisymmetric tubular reactor.

Figure 2: Mass of H₂ desorbed by the MeH material (dashes) and the CxH material (solid line) for a desorption scenario at 170 °C using a complete desorption in 3 h, (compare Table 1, scenario: A, D, E, F). ξ varies from 0.5 (green) to 1.25 (opt, black), 2 (blue) and 4 (turquoise).

Figure 3: Desorption scenario at 170 °C with ξ = 1.25 and d_{MeH} varies from 5 mm (orange) to 10 mm (black) and 15 mm (red), (compare Table 1, scenario: A, G, H). Mass of hydrogen desorbed by the MeH (dashes) and the CxH (solid) in kg per m length of reactor.

Figure 4: Effect of different module sizes on the flexibility of an overall hydride tank system during desorption.

Figure 5: Effective reaction rates versus transformed fraction for 165 °C and 0.5 bar (solid) or 1 bar (dashed line). A: Reference CxH material: 2 LiNH_2 – 1.1MgH_2 – 0.1 LiBH_4 –3wt.% ZrCoH₃ B: Reference MeH material: LaNi_{4.3}Al_{0.4}Mn_{0.3}.

Figure 6: Desorption scenario at 170 °C using the optimal absorption geometry and r_{des} for a complete desorption in 2 h (red), 3 h (black) and 5 h (blue), (compare Table 1, scenario: A, B, C). A: Mass of H₂ desorbed by the CxH (solid) and MeH (dashes). B: Temperature T_1 (solid) and T_2 (dashes).

Figure 7: Discharging rates for the three different scenarios together with the desorption rate versus the transformed fraction of the reference CxH material. Complete desorption in 3 h (solid, blue), varying load varying by a factor of 10 between low and high load (dash, red) and varying load varying by factor of 15 (dot, green). Arrows indicate max. possible transformed fraction for each maximum rate.

Figure 8: Desorption scenario for the three different scenarios: Complete desorption in 3 h (solid, blue), varying load by a factor of 10 between low and high load (dash, red) and varying load by a factor of 15 (dot, green). A: system pressure. B: mass H_2 desorbed by MeH material (turquoise, orange, light green) and mass H_2 desorbed by CxH material (blue, red and green).

Figure 9: Detail of Scenario II. System pressure (red, solid), desorption equilibrium pressure of the CxH (black, dot), desorption equilibrium pressure of the MeH (grey, dot) and absorption equilibrium pressure of the MeH (grey, dash).

Figure 10: Detail of mass H₂ desorbed by Meh material for the desorption scenarios shown in Figure 8. Turquoise: Scenario I, orange: Scenario II, light green: Scenario III.

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Tables

Table 1: Overview of the simulated geometries / scenarios.

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	Scenario	d_{MeH}	d _{s.s}	٤	ϵ_{CxH}	$t_{ m dis}$	$m_{ m MeH}$	m _{cxH}
		mm	mm	-	-	h	g per m length	g per m length
variation in t _{dis}	А	10	2.0	1.25	0.80	3	10.5	12.6
	В	10	2.0	1.25	0.80	2	10.5	10.1
	С	10	2.0	1.25	0.80	5	10.5	16.4
variation in දි	D	10	1.4	0.50	0.56	3	10.5	13.6
	E	10	2.7	2.00	0.89	3	10.5	11.3
	F	10	4.5	4.00	0.96	3	10.5	7.5
variation in d _{MeH}	G	5	1.6	2.50	0.92	3	2.6	8.2
	Н	15	2.5	0.83	0.70	3	23.5	16.5

Table 1: Overview of the simulated geometries / scenarios.

Figure 1: Scheme of the modelled 1D geometry of an axisymmetric tubular reactor.











Figure 4:



Figure 5:



Figure 6:







Figure 8:



Figure 9:





