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# A hierarchical finite element method based on the partition of unity

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## Abstract

In this paper we consider the application of hierarchical functions to base approximations which are a partition of unity. The particular hierarchical functions used are added to base finite element interpolations which, for  $C^0$  approximations, are a particular case of the partition of unity. We also show how the functions may be constructed to preserve the interpolation property of the base finite element functions. An application to linear elasticity is used to illustrate the properties and stability of the approximation.

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## 1. Introduction

The use of hierarchical interpolations for finite element analyses where approximations must be only  $C^0$  continuous usually adds functions to the edges, faces and interior of low order isoparametric interpolations (see e.g. [1]). For example, using a simple triangular element with linear interpolations quadratic and higher order functions may be added to each edge in a direct manner. To create complete polynomial approximations of cubic or higher order over the triangle it is necessary to also add an interior term. In recent publications [2–5], it has been shown that hierarchical functions may be added to any base approximation which is a partition of unity [6] and recover globally the base approximation *and* the added functions. Using this approach it is possible to have polynomial orders which vary from point to point in a mesh. Moreover, the addition of each polynomial term can be associated with nodes of the base approximation, without a need to add different types of functions for edges, faces and/or interior points. Thus, programming complexity for the approach can be significantly reduced.

In this paper we investigate the properties of the new hierarchical functions in the solution of any finite element problem for which  $C^0$  interpolations may be employed. In Section 2, we show how the functions may be given in a form which does not destroy the original interpolation capability of the base interpolations. The modified form for interpolation is shown to preserve the property of high order global Cartesian polynomial approximation while being straightforward to implement.

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In Section 3, we illustrate the basic structure for application of the method to plane strain, linear elasticity problems. Patch test evaluations, as described in [1,7] are used to assess the stability of the procedure using the simplest quadratic elements on triangles and quadrilaterals. Certain forms of the interpolation are found to be stable while others are not. Finally, for the triangular element form the method is applied and compared to classical isoparametric 6-node triangular element solutions.

## 2. Hierarchical interpolation

The basic form of the hierarchical approximation in one dimension is

$$\phi(x) = \sum_{a=1}^N N_a^k(x) [\bar{\phi}_a + \mathbf{P}^k(x) \bar{\mathbf{b}}_a] \quad (2.1)$$

with

$$\mathbf{P}^k(x) = [p_1(x) \quad p_2(x) \quad \dots] \quad (2.2)$$

and

$$\sum_{a=1}^N N_a^k(x) x_a^j = x^j \quad (2.3)$$

for  $j = 0, 1, \dots, k$  and any  $x$ . Note that for  $j$  zero the above implies that

$$\sum_{a=1}^N N_a^k(x) = 1 \quad (2.4)$$

which defines a partition of unity and also is required for  $C^0$  finite element approximation functions [1,6]. Thus, any  $C^0$  finite element interpolation function constitutes a basis for constructing the hierarchical approximation given by Eq. (2.1).

In the present work we modify Eq. (2.1) to

$$\phi(x) = \sum_{a=1}^N N_a^k(x) [\bar{\phi}_a + \mathbf{P}_a^k(x) \bar{\mathbf{b}}_a] \quad (2.5)$$

and use polynomial approximation for the  $\mathbf{P}_a^k$ .

The first derivative of Eq. (2.5) in each element is given by

$$\frac{d\phi}{dx} = \sum_{a=1}^N \left\{ \frac{dN_a^k}{dx} \bar{\phi}_a + \left[ \frac{dN_a^k}{dx} \mathbf{P}_a^k + N_a^k \frac{d\mathbf{P}_a^k}{dx} \right] \bar{\mathbf{b}}_a \right\} \quad (2.6)$$

The following hierarchical forms are used for the work reported here:

*Form A in which the polynomials are constructed as*

$$\mathbf{P}_a^k(x) = [(x - x_a)^{k+1} \quad (x - x_a)^{k+2} \quad \dots] \quad (2.7)$$

or

*Form B in which the polynomials are constructed as*

$$\mathbf{P}_a^k(x) = [(x^{k+1} - x_a^{k+1}) \quad (x^{k+2} - x_a^{k+2}) \quad \dots] \quad (2.8)$$

Note that the hierarchical functions are also now associated to each node,  $a$ .

### 2.1. Form A: One dimensional approximations

To illustrate the performance of the approximation given by Eq. (2.5), we consider a one-dimensional element with two nodes and take

$$x = \sum_{a=1}^2 N_a^1(\xi) \bar{x}_a \quad (2.9)$$

where  $\xi$  is a natural coordinate satisfying

$$-1 \leq \xi \leq 1$$

and  $N_a^1$  are shape functions in each element given by

$$N_1^1(\xi) = \frac{1}{2}(1 - \xi); \quad N_2^1(\xi) = \frac{1}{2}(1 + \xi) \quad (2.10)$$

For a quadratic polynomial approximation use the form given by Eq. (2.7) with

$$P_a^1(x) = (x - x_a)^2 \quad (2.11)$$

Thus, within each element the approximation is given by

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) [\bar{\phi}_a + (x - x_a)^2 \bar{b}_a] \quad (2.12)$$

Let us now verify that the approximation contains the polynomials

$$\phi(x) = [1 \quad x \quad x^2]$$

(a) Set  $\bar{b}_a$  to zero and  $\bar{\phi}_a$  to unity. We obtain

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) = 1 \quad (2.13)$$

(b) Set  $\bar{b}_a$  to zero and  $\bar{\phi}_a$  to  $x_a$ . We obtain

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) x_a = x \quad (2.14)$$

(c) Set  $\bar{b}_a$  to  $-1$  and  $\bar{\phi}_a$  to  $x_a^2$ . We obtain

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) [x_a^2 - (x - x_a)^2]$$

which expands to

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) [2x x_a^2 - x^2]$$

and by Eq. (2.3) becomes

$$\phi(x) = x^2 \quad (2.15)$$

Joining elements to form an arbitrary patch preserves the above properties for global polynomials. Thus, the polynomial form given by Eqs. (2.12) *interpolates* through points located at  $x_a$  and contains complete *global* polynomials to order 2.

## 2.2. Form B: One-dimensional approximations

Form B may also be used to perform the interpolations. Accordingly, for the one-dimensional case with quadratic polynomial approximation we introduce Eq. (2.8) as

$$\mathbf{P}_a^1(x) = (x^2 - x_a^2) \quad (2.16)$$

Thus, within each element the approximation for this form is given by

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) [\bar{\phi}_a + (x^2 - x_a^2) \bar{b}_a] \quad (2.17)$$

Let us now verify that this approximation contains the polynomials

$$\phi(x) = [1 \quad x \quad x^2]$$

(a) Set  $\bar{b}_a$  to zero and  $\bar{\phi}_a$  to unity. We obtain

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) = 1 \quad (2.18)$$

(b) Set  $\bar{b}_a$  to zero and  $\bar{\phi}_a$  to  $x_a$ . We obtain

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) x_a = x \quad (2.19)$$

(c) Set  $\bar{b}_a$  to 1 and  $\bar{\phi}_a$  to  $x_a^2$ . We obtain

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) [x_a^2 + (x^2 - x_a^2)]$$

which simplifies to

$$\phi(x) = \sum_{a=1}^2 N_a^1(x) x^2$$

and using Eq. (2.3) becomes

$$\phi(x) = x^2 \quad (2.20)$$

Joining elements to form an arbitrary patch preserves the above properties for global polynomials. Thus, the polynomial form given by Eq. (2.17) also *interpolates* through points located at  $x_a$  and contains complete *global* polynomials to order 2.

It is evident that the modification given by Eq. (2.5) also makes development of finite elements much simpler than using Eq. (2.1) since the nodal parameters  $\bar{\phi}_a$  retain their original finite element physical meaning. Furthermore, the specification of essential and natural boundary conditions may directly use the interpolation property.

## 2.3. Two-dimensional approximations

For two-dimensional problems, the approximations in each element may be taken as

$$\mathbf{x}(\xi) = \sum_{a=1}^N N_a^k(\xi) \bar{\mathbf{x}}_a \quad (2.21a)$$

for coordinates and

$$\phi(\xi) = \sum_{a=1}^N N_a^k(\xi) [\bar{\phi}_a + \mathbf{P}_a^k(\mathbf{x}(\xi)) \bar{\mathbf{b}}_a] \quad (2.21b)$$

for dependent variables. In the above  $N$  is the number of nodes on each element and the  $\xi$  are appropriate natural coordinates.

2.3.1. Three node triangular element

The simplest element in two dimensions is a triangle with three nodes where  $\xi$  are area coordinates,  $\xi_1, \xi_2, \xi_3$  which satisfy

$$0 \leq \xi_i \leq 1$$

and

$$\xi_1 + \xi_2 + \xi_3 = 1$$

The shape functions for the triangle are given as

$$N_a^1 = \xi_a \tag{2.22}$$

The hierarchical terms for Form A are similar to Eq. (2.7) and here are taken as

$$\mathbf{P}_a^1(\mathbf{x}(\xi)) = [(x - x_a)^2 \quad (x - x_a)(y - y_a) \quad (y - y_a)^2] \tag{2.23}$$

with parameters

$$\bar{\mathbf{b}}_a = \begin{bmatrix} \bar{b}_a^{xx} \\ \bar{b}_a^{xy} \\ \bar{b}_a^{yy} \end{bmatrix} \tag{2.24}$$

The approximation for Form B becomes

$$\mathbf{P}_a^1(\mathbf{x}(\xi)) = [x^2 - x_a^2 \quad (xy - x_a y_a) \quad (y^2 - y_a^2)] \tag{2.25}$$

and has similar parameter structure to Form A.

The approximation produces an arbitrary element patch which contains all the quadratic order terms. The proof is identical to the one-dimensional case given above.

The first partial derivatives of Eq. (2.21) in each element are given by

$$\frac{\partial \phi}{\partial x} = \sum_{a=1}^N \left\{ \frac{\partial N_a^k}{\partial x} \bar{\phi}_a + \left[ \frac{\partial N_a^k}{\partial x} \mathbf{P}_a^k + N_a^k \frac{\partial \mathbf{P}_a^k}{\partial x} \right] \bar{\mathbf{b}}_a \right\} \tag{2.26a}$$

and

$$\frac{\partial \phi}{\partial y} = \sum_{a=1}^N \left\{ \frac{\partial N_a^k}{\partial y} \bar{\phi}_a + \left[ \frac{\partial N_a^k}{\partial y} \mathbf{P}_a^k + N_a^k \frac{\partial \mathbf{P}_a^k}{\partial y} \right] \bar{\mathbf{b}}_a \right\} \tag{2.26b}$$

For the quadratic polynomial case we have, for Form A, the result

$$\frac{\partial \mathbf{P}_a^k}{\partial x} = [2(x - x_a) \quad (y - y_a) \quad 0] \tag{2.27a}$$

and

$$\frac{\partial \mathbf{P}_a^k}{\partial y} = [0 \quad (x - x_a) \quad 2(y - y_a)] \tag{2.27b}$$

whereas for Form B the result is

$$\frac{\partial \mathbf{P}_a^k}{\partial x} = [2x \quad y \quad 0] \tag{2.28a}$$

and

$$\frac{\partial \mathbf{P}_a^k}{\partial y} = [0 \quad x \quad 2y] \tag{2.28b}$$

For problems in which the coordinates attain large values, Form A leads to better numerical conditioning in the evaluation of integrals. For the quadratic interpolation case, we have for Form B the result

$$\phi(\xi) = \sum_{a=1}^N N_a^k(\xi) [\hat{\phi}_a + [x^2, xy, y^2] \bar{\mathbf{b}}_a] \quad (2.29)$$

where

$$\hat{\phi}_a = \bar{\phi}_a - x_a^2 \bar{b}_a^{xx} - x_a y_a \bar{b}_a^{xy} - y_a^2 \bar{b}_a^{yy} \quad (2.30)$$

Expansion of Form A produces results which include additional terms involving the linear polynomials  $x$  and  $y$ . Hence, Form B merely changes the definition of the parameters defining the original finite element interpolations whereas Form A changes the structure of the original interpolation. Both forms, however, are legitimate quadratic hierarchical interpolations.

Other interpolation types and orders may be obtained by changing the dimension of the problem, the order of the basic approximations  $N_a^k$ , and/or the order of the terms included in  $\mathbf{P}_a^k$ . For example, if the basic approximations are  $N_a^2$  (i.e. contain all polynomials up to order 2) a cubic interpolation for Form A may be constructed using Eq. (2.5) with

$$\mathbf{P}_a^2(x) = [(x - x_a)^3 \quad (x - x_a)^2(y - y_a) \quad (x - x_a)(y - y_a)^2 \quad (y - y_a)^3] \quad (2.31)$$

Similar constructions may also be performed for two-dimensional quadrilateral elements or three-dimensional tetrahedra or brick elements. As noted in [8], however, we must ensure that no linear dependencies occur in the final interpolation form.

It is possible to introduce other types of functions for the  $\mathbf{P}_a^k$  as indicated in [2] and [5].

### 3. Application to linear elasticity

In the previous section the form for polynomial interpolation in two dimensions was given. Here, we consider the use of the interpolation to model behavior of problems in linear elasticity. For simplicity only displacement models are considered; however, the use of the hierarchic interpolations can easily be extended to mixed models.

For a displacement model the finite element arrays may be deduced from the principal of minimum potential energy given here by

$$\Pi(\mathbf{u}) = \int_{\Omega} W(\boldsymbol{\epsilon}) \, d\Omega - \int_{\Omega} \mathbf{u}^T \mathbf{b} \, d\Omega - \int_{\Gamma_t} \mathbf{u}^T \bar{\mathbf{t}} \, d\Omega \quad (3.1)$$

where  $\mathbf{u}$  are displacement components,  $\boldsymbol{\epsilon}$  are strains,  $W$  the strain energy,  $\mathbf{b}$  body forces, and  $\bar{\mathbf{t}}$  specified boundary tractions. The strain–displacement relations are given by

$$\boldsymbol{\epsilon} = \nabla^{(s)} \mathbf{u} \quad (3.2)$$

where  $\nabla^{(s)}$  denotes the symmetric part of the displacement gradient. For two-dimensional plane strain problems, the strain–displacement relations may be written in matrix form as

$$\boldsymbol{\epsilon} = \begin{bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{bmatrix} \quad (3.3)$$

Inserting the interpolations for  $u$  and  $v$  given by Eq. (2.5), the strain–displacement relations become

$$\epsilon = \sum_{a=1}^N \begin{bmatrix} \frac{\partial N_a^k}{\partial x} & 0 \\ 0 & \frac{\partial N_a^k}{\partial y} \\ \frac{\partial N_a^k}{\partial y} & \frac{\partial N_a^k}{\partial x} \end{bmatrix} \begin{bmatrix} \bar{u}_a \\ \bar{v}_a \end{bmatrix} + \sum_{a=1}^N \begin{bmatrix} \left( \frac{\partial N_a^k}{\partial x} \mathbf{P}_a^k + N_a^k \frac{\partial \mathbf{P}_a^k}{\partial x} \right) & 0 \\ 0 & \left( \frac{\partial N_a^k}{\partial x} \mathbf{P}_a^k + N_a^k \frac{\partial \mathbf{P}_a^k}{\partial y} \right) \\ \left( \frac{\partial N_a^k}{\partial x} \mathbf{P}_a^k + N_a^k \frac{\partial \mathbf{P}_a^k}{\partial y} \right) & \left( \frac{\partial N_a^k}{\partial x} \mathbf{P}_a^k + N_a^k \frac{\partial \mathbf{P}_a^k}{\partial x} \right) \end{bmatrix} \begin{bmatrix} \bar{b}_a^u \\ \bar{b}_a^v \end{bmatrix} \quad (3.4)$$

The first term is identical to usual finite element strain–displacement matrices (see e.g. [1]) and the second term has identical structure to the usual arrays. Thus, the development of all element arrays follows standard procedures.

### 3.1. Quadratic triangular element

For a triangular element with linear interpolation the shape functions and quadratic polynomial hierarchic terms are given by Eqs. (2.22) and (2.23) or (2.25), respectively. Using isoparametric concepts the coordinates are given by

$$\mathbf{x} = \sum_{a=1}^3 N_a^1 \bar{\mathbf{x}}_a = \sum_{a=1}^3 \xi_a \bar{\mathbf{x}}_a \quad (3.5)$$

and are used to construct all polynomials appearing in the hierarchical form.

#### Patch test

A set of patch tests is first performed to assess the stability and consistency of the above hierarchic form. The set consists of one, two, four, and eight element patches as shown in Fig. 1. First, we perform a stability assessment by determining the number of zero eigenvalues for each patch. The results for hierarchical interpolation using Form A are shown in Table 1 and for Form B in Table 2.

The eigenproblem assessment reveals that the hierarchic interpolation using Form A has excess zero eigenvalues (i.e. spurious zero energy modes) only for meshes consisting of one or two elements. Furthermore, only two element meshes in which one side is a straight line through both elements have excess

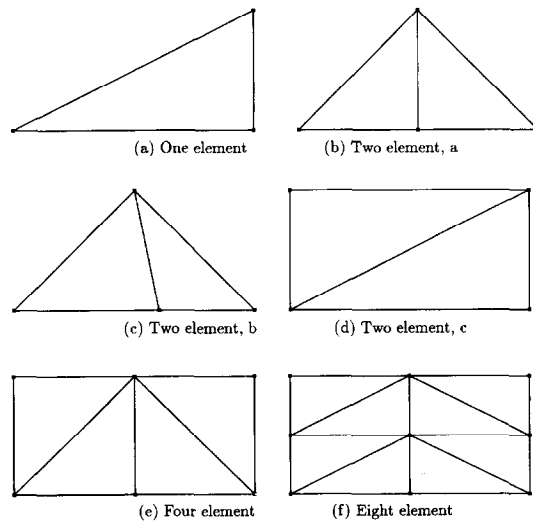


Fig. 1. Patches for eigenproblem assessment.

Table 1

Triangle patch tests for Form A: Number zero eigenvalues, minimum non-zero value, and maximum value

Mesh	No. zero	Min. value	Max. value
1	7	4.7340E+01	2.0560E+06
2a	5	4.0689E+01	2.1543E+05
2b	5	4.1971E+02	2.2648E+05
2c	3	1.5728E+02	2.3883E+06
4	3	1.0446E+02	2.9027E+05
8	3	9.5560E+01	3.4813E+05

Table 2

Triangle patch tests for Form B: Number zero eigenvalues, minimum non-zero value, and maximum value

Mesh	No. zero	Min. value	Max. value
1	7	4.6067E+01	1.0120E+07
2a	7	4.5559E+01	1.8880E+06
2b	7	4.3815E+01	2.2254E+06
2c	7	1.6340E+02	1.0690E+07
4	7	1.0847E+02	3.7646E+06
8	7	8.8303E+01	5.3064E+06

zero values. Once the mesh has no straight intersections the number of zero modes becomes correct (e.g. contain only the three rigid body modes). On the other hand, using Form B spurious zero modes persist for all meshes. We note that Form B involves only a simple change of the definition for the finite element parameters, and thus the spurious results obtained in Table 2 also apply for the interpolation given by Eq. (2.1).

Consistency tests verify that all meshes contain terms of up quadratic polynomial order.

### 3.2. Quadratic quadrilateral element

For a quadrilateral element with bi-linear interpolation the shape functions given by

$$N_a^1(\xi, \eta) = \frac{1}{4}(1 + \xi_a \xi)(1 + \eta_a \eta) \quad (3.6)$$

where  $\xi, \eta$  are natural coordinates satisfying

$$-1 \leq \xi, \eta \leq 1$$

and  $\xi_a, \eta_a$  are the values of the natural coordinates at vertex nodes. Using isoparametric interpolations the coordinates may be computed from

$$\mathbf{x} = \sum_{a=1}^4 N_a^1(\xi, \eta) \mathbf{x}_a \quad (3.7)$$

Quadratic polynomial hierarchic terms are constructed for Form A using Eq. (2.23).

#### 3.2.1. Patch test

A set of patch tests is first performed to assess the stability and consistency of the above hierarchic form on rectangles. The set consists of one, two and four element patches as shown in Fig. 2. First, we perform a stability assessment by determining the number of zero eigenvalues for each patch. The results are shown in Table 3.

The eigenproblem assessment reveals that the hierarchic interpolation using Form A has excess zero eigenvalues (i.e. spurious zero energy modes) for all meshes considered. Further considerations reveal that the use of quadratic polynomials for  $P$  which include  $xy$  introduce linearly dependent terms since



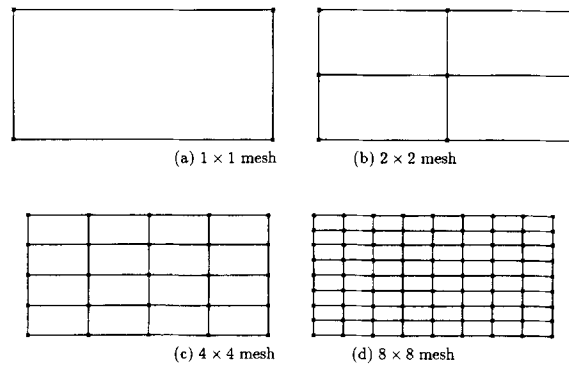


Fig. 2. Patches for rectangle eigenproblem assessment.

this term is already available when the elements have constant jacobian. Repeating the stability test after eliminating this polynomial yields the results given in Table 4.

Repeating the above for Form B yields the results in Tables 5 and 6. The eigenproblem assessment reveals that the hierarchic interpolation using Form B also has excess zero eigenvalues for all meshes considered; however, eliminating the polynomial  $xy$  again restores proper rank as shown in Table 6.

Consistency tests again confirm the existence of all polynomial terms up to quadratic order.

Table 3

Quadrilateral patch tests for Form A: Number zero eigenvalues, minimum non-zero value, and maximum value

Mesh	No. zero	Min. value	Max. value
1 × 1	9	1.5577E+02	8.6894E+05
2 × 2	13	9.6509E+01	1.3080E+05
4 × 4	21	3.7956E+00	1.7859E+04
8 × 8	37	2.4424E-02	9.7290E+03

Table 4

Quadrilateral patch tests Form A: Number zero eigenvalues, minimum non-zero value, and maximum value

Mesh	No. zero	Min. value	Max. value
1 × 1	3	1.5739E+02	8.6894E+05
2 × 2	3	9.5719E+01	1.3073E+05
4 × 4	3	4.8047E+01	1.7859E+04
8 × 8	3	4.3963E+00	9.7290E+03

Table 5

Quadrilateral patch tests for Form B: Number zero eigenvalues, minimum non-zero value, and maximum value

Mesh	No. zero	Min. value	Max. value
1 × 1	9	1.5498E+02	4.8936E+06
2 × 2	9	4.8788E+01	3.3575E+06
4 × 4	9	4.4192E-01	1.5102E+06
8 × 8	9	3.0824E-03	5.2649E+05

Table 6

Quadrilateral patch tests for Form B: Number zero eigenvalues, minimum non-zero value, and maximum value

Mesh	No. zero	Min. value	Max. value
1 × 1	3	1.5599E+02	4.4176E+06
2 × 2	3	8.7079E+01	3.2108E+06
4 × 4	3	8.1631E+00	9.0948E+05
8 × 8	3	4.8673E-01	4.1174E+05

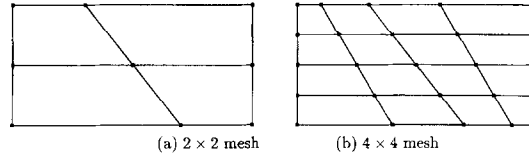


Fig. 3. Patches for quadrilateral eigenproblem assessment.

The mesh for rectangles was distorted as shown in Fig. 3. With the quadratic polynomial term  $xy$  retained, the spurious eigenvalues persist, and with it removed the consistency test is failed. Thus, the quadrilateral element does not satisfy patch requirements, and in the present work is not considered further.

### 3.3. Tension strip

A finite width strip containing a circular hole with diameter half the width of the strip is subjected to axial extension in the vertical direction. Due to symmetry of the loading and mesh, only one quadrant is discretized. A set of meshes is shown in Figs. 4 and 5. Meshes in Fig. 4 use Form A of the hierarchical interpolation considered above; whereas those in Fig. 5 use standard 6-node isoparametric quadratic triangles with two degrees of freedom per node (i.e.  $u$  and  $v$ ).

The problem size and solution statistics are shown in Table 7 for the hierarchical Form A method and in Table 8 for the 6-node isoparametric formulation where mid-side element nodes are permitted to lie on the curved boundary. In Table 9, we present the results when the mid-nodes are linearly interpolated from the vertex nodes (which produces the same boundary interpolation as the new hierarchical form).

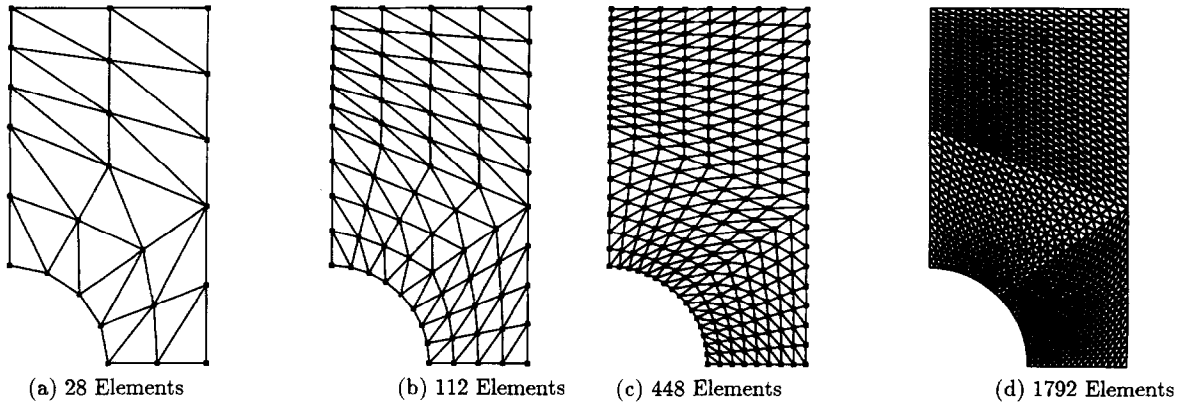


Fig. 4 Hierarchic elements form A: Tension strip.

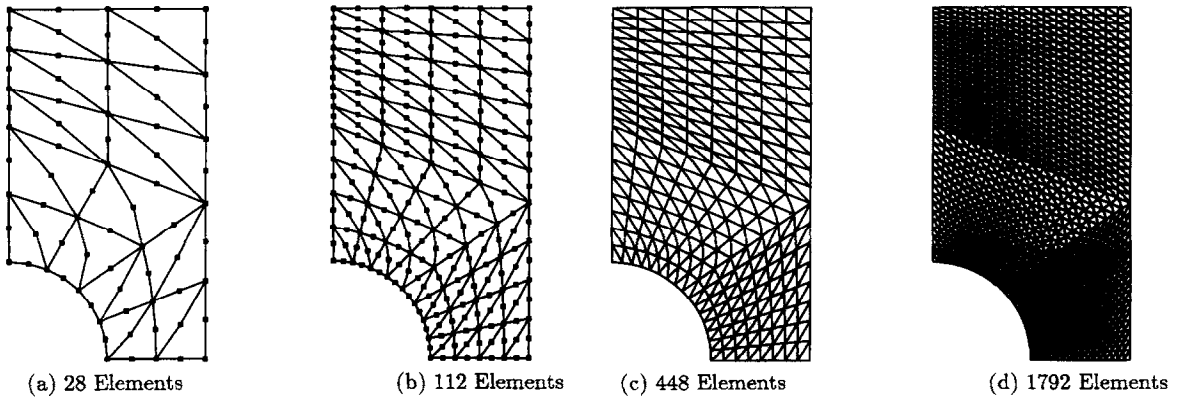


Fig. 5. Isoparametric elements: Tension strip.

Table 7  
Hierarchical element problem size statistics

Nodes	Elements	Equations	Non-zeros	Profile	Column	Energy
30	28	156	2892	4159	28	131.7088
85	112	537	12015	29860	56	127.8260
279	448	1971	48741	207508	106	126.7641
1003	1792	7527	196113	1547658	205	126.5908

Table 8  
Isoparametric element problem size statistics. Boundary segments have curved sides

Nodes	Elements	Equations	Non-zeros	Profile	Column	Energy
30	28	129	1070	1777	14	127.3350
279	112	483	4613	13428	28	126.6483
1003	448	1863	19091	100970	55	126.5661
3795	1792	7311	77615	769110	106	126.5593

Table 9  
Isoparametric element problem size statistics. Boundary segments have straight sides

Nodes	Elements	Equations	Non-zeros	Profile	Column	Energy
30	28	129	1070	1777	14	128.6458
279	112	483	4613	13428	28	126.9451
1003	448	1863	19091	100970	55	126.6090
3795	1792	7311	77615	769110	106	126.5394

Table 10  
Linear triangular element problem size statistics

Nodes	Elements	Equations	Non-zeros	Profile	Column	Energy
30	28	36	150	252	7	137.652
85	112	129	669	1711	13	131.065
279	448	483	2799	12438	26	128.008
1003	1792	1863	11427	94696	50	126.958
3795	7168	7311	46155	738972	102	126.662

Finally, in Table 10 we present the results using 3-node linear triangular elements. For the analyses, the material is taken as linear elastic with  $E = 1000$  and  $\nu = 0.25$ . The half width of the strip is 10 units and the half height is 18 units. The hole has radius 5.

The best results obtained are consistently for the 6-node isoparametric element. The effect of boundary error is negligible for this problem, especially for the finer meshes. The new element, however, is consistently better than the 3-node triangular element. Consequently, the potential of the new interpolation for adaptive refinement should be evaluated.

#### 4. Closure

In this work we have discussed the use of hierarchical approximations which are appended to conventional  $C^0$  finite element interpolations. The form is based on adding polynomial functions to a partition of unity. The particular form considered permits polynomials to any degree to be connected around any node in a mesh. Thus, the method is particularly amenable to adaptive processes. Furthermore, we have shown how the polynomials may be constructed to preserve the interpolating property of the original finite element based functions. For linear elasticity problems, as well as for any form requiring  $C^0$  approximations, the interpolation preserves the structure of basic finite element strain–displacement matrices and, thus, permits a simple construction of basic arrays.

Based on patch test assessments for the linear elasticity problem, we have demonstrated that some forms are stable, while others are not. In particular, triangles based on Form A achieve stability provided

more than two elements are used; whereas, Form B interpolation never achieved stability for all boundary conditions. The case of rectangular elements achieved stability provided the interpolation was constructed to avoid linear dependencies. However, the results for quadrilateral elements indicated that neither Form A nor Form B could produce interpolations which were both stable and consistent to quadratic polynomial order. Hence, further study is necessary to achieve success for this class of elements.

Finally, we have compared the formulation for meshes composed of elements with quadratic accuracy at all nodes to conventional 6-node isoparametric triangles which for straight sides also achieve quadratic accuracy. Here, results favor the 6-node quadratic triangles: both for efficiency in overall solution effort, as well as for accuracy based on energy. When compared to 3-node triangular elements the new interpolation shows considerable improvement in accuracy; consequently, further tests in an adaptive environment are needed before a more complete assessment may be reached.

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