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VIBRATION-BASED HEALTH MONITORING APPROACH FOR COMPOSITE STRUCTURES USING MULTIVARIATE STATISTICAL ANALYSIS.

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ABSTRACT

In this paper a novel procedure for damage assessment is suggested which is based on singular spectrum analysis (SSA). The main feature of the method is that it applies Principal Component Analysis (PCA) to the lagged time series, obtained from the measured structural vibration response. In this study the methodology is developed for the case of a free decay response. The measured acceleration vectors are transformed into the frequency domain and then used to define a trajectory matrix. The covariance matrix of the trajectory matrix is decomposed into new variables, the Principal Components (PCs). They define a new space of linearly correlated variables onto which the dynamics/motion of the system can be projected. This decomposition is used to uncover oscillation patterns among other purposes. The method is applied and demonstrated for the case of a simple 2-DoF system. To demonstrate its capabilities for damage diagnosis different levels of stiffness reduction are introduced. The first two PCs are used to visually demonstrate the abilities of the methodology. The Mahalanobis distance is used to develop a classification system to detect and localize delamination in the 2-DoF system. The results clearly demonstrate the capabilities of the system to clearly detect and localize damage.

KEYWORDS : *Composite Materials, Delamination, Singular Spectrum Analysis, Statistical Pattern Recognition, Damage assessment*

INTRODUCTION

The growth in aerospace and other sectors of engineering placed composite materials at the forefront of the contemporary research. Modern structures typically require the use of lightweight and strong materials, which demand high level of performance combined with greater efficiency. Composites are generally implemented in structures with tremendous engineering requirements and their nature depends on the material composition. This presents one of the biggest advantages of composites, they combine the advantageous properties of two or more materials. The design, the analysis and the production of such materials require a high level of knowledge and expertise in different engineering areas. However, composites also have some disadvantages, delamination. Damage and delamination in composite structures affects adversely the system's performance which makes Structural Health Monitoring (SHM) a must for such materials and structures. The development of a proper structural health monitoring system has a crucial importance for such structures because they are impossible to repair and very difficult to monitor for defects.

Vibration-based Structural Health Monitoring (VSHM) is a methodology which extracts information from the vibratory response of the structures. In this particular case it was used to give an assessment about the presence, localization and severity of the damage in structures [1]. VSHM methods can be divided in two main groups: model based and non-model based ones. This paper develops a new non-model based methodology for damaged assessment which uses the time-domain structural vibration signals in order to model and/or analyse the structural dynamic behaviour and to derive conclusions about the structure's health and integrity. Such methods draw their roots from the structural

non-linear dynamics and they utilise Takens theorem [2] according to which, any dynamic system can be fully reconstructed in a space made of lagged components of its measured time series. Thus, if one is able to measure a time dependent variable from a vibrating structure e.g its displacement or acceleration, the lagged components of this time series can be used to make a new space in which its behaviour can be analysed and ultimately reconstructed.

In practice most purely data-based methodologies make use of data analysis and utilise different statistical methods and characteristics. One of those methods is Principal Component Analysis (PCA) which is utilized for compressing and reducing the dimension of the data. PCA is used as an optimal linear tool and it have been extensively utilised in structural dynamics for modal analysis [3], reduce-order modelling [4], model updating of non-linear systems [5] and damage detection [6]. However PCA assumes that original measured variables are uncorrelated whilst time series elements like acceleration or displacement in discrete time are generally correlated. That is why in this paper an extension of PCA, known as Singular Spectrum Analysis (SSA) is studied [7], which is particularly developed for non-independent data. SSA can be applied in time and frequency domain [8] where the data is better organised. SSA can be applied for linear as well as for non-linear vibratory systems [2]. SSA uses the same procedure as PCA methodologies which project the original data onto a new space with smaller dimension. This new space is made of the lagged signal components. Differently from traditional spectrum analysis, SSA is able to uncover rotational periodicities at any frequency. Thus, in a certain sense it can be applied for the purpose of modal analysis for non-linear vibrating structures [9].

By using a certain number of components, a reconstruction of the original signal can be obtained with a very high accuracy. The original signal is projected onto the space created by reconstructed components (RCs) by selecting the first several RCs with the highest variance. PCA can be also used for classification purposes, thus when there are different categories in the measured data PCA is likely to facilitate the distinction among these. In this study, SSA and the Mahalanobis distance is combined to develop a classifier for the purposes of damage detection and localization in a 2-DoF system. The combination of the two methodologies demonstrates the capabilities of the system to clearly detect and localize damage.

1. SINGULAR SPECTRUM ANALYSIS FOR DAMAGE ASSESSMENT

In this section SSA is introduced with relation to structural dynamics and damage assessment. The suggested methodology for damage assessment is derived and explained.

1.1 Methodology

The *data collected* was made up from the acceleration response measured on a simulated system. Multiple realisations of the acceleration response were measured and gathered into a vector $\mathbf{x}^i = (\mathbf{x}_1^i, \mathbf{x}_2^i, \dots, \mathbf{x}_j^i, \dots, \mathbf{x}_N^i)'$ where $i = 1, 2, \dots, M$ is the number of realisations and $j = 1, 2, \dots, N$ is the number of components in each signal.

The acceleration response signal corresponding to each realisation was transformed into the frequency domain. In this way the spectral data matrix $\mathbf{Z} = (\mathbf{z}^1, \mathbf{z}^2, \dots, \mathbf{z}^i, \dots, \mathbf{z}^M)$ was obtained.

The next step is to *embed* the vibratory responses which define the trajectory of the new space.

A W -dimensional space was created in which the frequency domain vectors were embedded [8]. In this way a vector \mathbf{z}^i was embedded into (and substituted by) the matrix showed bellow.

$$\check{\mathbf{Z}}^i = \begin{pmatrix} z_1^i & z_2^i & z_3^i & \cdots & z_w^i & \cdots & z_W^i \\ z_2^i & z_3^i & z_4^i & \cdots & z_{w+1}^i & \cdots & z_{W+1}^i \\ z_3^i & z_4^i & z_5^i & \cdots & z_{w+2}^i & \cdots & z_{W+2}^i \\ \vdots & \vdots & \vdots & \cdots & \vdots & \cdots & \vdots \\ z_4^i & z_5^i & z_6^i & \cdots & z_{w+3}^i & \cdots & \vdots \\ z_5^i & z_6^i & \vdots & \cdots & \vdots & \cdots & z_{N'}^i \\ z_6^i & \vdots & \vdots & \cdots & z_{N'}^i & \cdots & 0 \\ \vdots & \vdots & z_{N'}^i & \cdots & 0 & \cdots & 0 \\ \vdots & z_{N'}^i & 0 & \cdots & 0 & \cdots & 0 \\ z_{N'}^i & 0 & 0 & \cdots & 0 & \cdots & 0 \end{pmatrix} \quad (1)$$

where $i = 1, 2, \dots, M$, $w = 1, 2, \dots, W$ and $N' = \frac{N}{2}$. All the matrices $\check{\mathbf{Z}}^i$ were collected to make a new matrix defined as $\check{\mathbf{Z}} = (\check{\mathbf{Z}}^1, \check{\mathbf{Z}}^2, \dots, \check{\mathbf{Z}}^i, \dots, \check{\mathbf{Z}}^M)$.

At the next step, the covariance matrix of the matrix $\check{\mathbf{Z}}$ was obtained following the Equation below.

$$\mathbf{C}_Z = \frac{\check{\mathbf{Z}}' \check{\mathbf{Z}}}{N'} \quad (2)$$

The eigenvalues λ_k and the eigenvectors ρ_k of \mathbf{C}_Z were obtained according to the following expression.

$$\mathbf{C}_Z \rho_k = \lambda_k \rho_k \quad (3)$$

The eigenvalues λ_k were then ordered in the diagonal matrix $\mathbf{\Lambda}_Z$ in decreasing order and the matrix \mathbf{E}_Z contains their corresponding eigenvectors ρ_k written as columns. The \mathbf{E}_Z vectors are called Empirical Orthogonal Functions (EOFs) and they contain the data as a *decomposition* into orthogonal basis. The eigenvalues define the partial variance of each eigenvectors, therefore the total sum of all of these variances gives the total variance of $\check{\mathbf{Z}}$.

$$\mathbf{E}_Z' \mathbf{C}_Z \mathbf{E}_Z = \mathbf{\Lambda}_Z \quad (4)$$

The measured data $\check{\mathbf{Z}}$ was projected onto the matrix \mathbf{E}_Z , which yields the corresponding Principal Components (PCs) matrix $\mathbf{A} = \check{\mathbf{Z}} \mathbf{E}_Z$.

A matrix which contains the projection of the PCs in to the new space was created to *reconstruct* the signal. The Reconstructed Components (RCs) were obtained according to the Equation (5). For a given set of indices \mathcal{K} corresponding to a set of PCs, the RCs were obtained by projecting the corresponding PCs onto the EOFs, as it is shown in the Equation 5.

$$R_{m,n}^k = \frac{1}{W} \sum_{w=1}^W A_{n-w}^k E_{m,w}^k \quad (5)$$

where k -eigenvectors give the k^{th} RC at n -frequency between $n = 1 \dots N'$ for each m -channel ($m = 1 \dots M$) which was embedded in a w -lagged vectors with the maximum W -length.

1.2 Projection. Clustering effect

The oscillatory responses of the system were decomposed using a certain number of RCs. Each one contains a certain percentage of variance provided by the EOFs. The projection of the original data onto the RCs can be modelled as a single point [10]. The points into the new space created by the orthogonal basis (EOFs) gave the coordinates of the projection of the original data \mathbf{Z} in the RCs. The coordinates characterize the vibratory responses on a point. The information contained within these projections was utilised as pattern recognition features for the classification purposes. The components containing more of the variance of the initial signal have more information. Therefore, they contain most of the information about the changes in the vibratory system. This projection was developed following the Equation 6.

$$\mathbf{T} = \langle \mathbf{Z}, \mathbf{R} \rangle \quad (6)$$

where \mathbf{Z} is the original frequency signal matrix and the matrix \mathbf{R} contains the reconstructed components.

1.3 Mahalanobis distance

The Mahalanobis distance measures the relative distance from a data set of points, denoted as observation data, to a common points, denoted as reference data. The Mahalanobis distance considers the correlation of the data set, therefore it can be perfectly used to multivariate data sets. In this particular study, the combination of the Mahalanobis distance and SSA methodology was developed. The projection obtained in §1.2 were used as pattern recognition features for the classification purposes. The use of the Mahalanobis distance offers the possibility to consider more components from the SSA decomposition. It can increase the information needed for the correct classification.

The Mahalanobis distance is described by the following Equation 7.

$$\mathbf{D}_M(T_R) = \sqrt{(\mathbf{T}_O - \mu_R)^T \mathbf{S}_R^{-1} (\mathbf{T}_O - \mu_R)} \quad (7)$$

where $\mathbf{D}_M(T_R)$ is the Mahalanobis distance to the reference data set \mathbf{T}_R , \mathbf{T}_O is the observation data, μ_R is the means of the \mathbf{T}_R and \mathbf{S}_R is the standard deviation of \mathbf{T}_R .

2. EXAMPLE APPLICATION: 2DOF NONLINEAR SPRING-MASS-DAMPER SYSTEM.

The evaluation of the methodology was applied in a 2-DoF system with a nonlinear spring connecting two mass, the schematic system is shown in the Figure 1 and it is described by the Equation 8

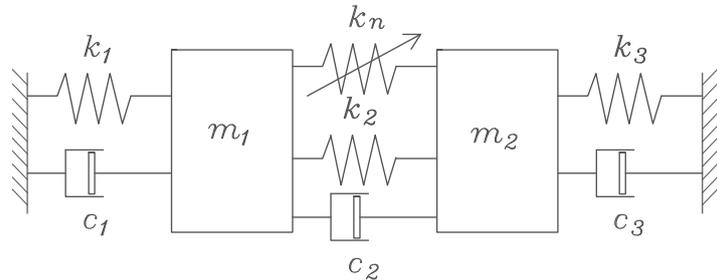


Figure 1 : 2-DoF spring-mass damper with non-linear stiffness system

$$[\mathbf{M}]\ddot{\mathbf{x}} + [\mathbf{C}]\dot{\mathbf{x}} + [\mathbf{K}]\mathbf{x} + \mathbf{f}(\dot{\mathbf{x}}, \mathbf{x}) = 0 \quad (8)$$

where $[\mathbf{M}]$, $[\mathbf{C}]$, $[\mathbf{K}]$ are constant coefficients mass, damping and stiffness matrices respectively defined by the Equation (9). The function $\mathbf{f}(\dot{\mathbf{x}}, \mathbf{x})$ provides a quadratic coupling between masses and it is defined by Equation (10).

$$[\mathbf{M}] = \begin{pmatrix} m_1 & 0 \\ 0 & m_2 \end{pmatrix} \quad [\mathbf{C}] = \begin{pmatrix} c_1 + c_2 & -c_2 \\ -c_2 & c_2 + c_3 \end{pmatrix} \quad [\mathbf{K}] = \begin{pmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{pmatrix} \quad (9)$$

$$\mathbf{f}(\dot{\mathbf{x}}, \mathbf{x}) = \begin{Bmatrix} -k_n(x_2 + x_1)^2 \\ k_n(x_2 + x_1)^2 \end{Bmatrix} \quad (10)$$

The value of the parameters corresponding to the initial conditions (without any stiffness reduction) are detailed in the Table 1. These values simulate the system as healthy. An initial velocity in the m_2 was applied to simulate an impulse. The system describe a free-decay response and the acceleration was recorded by each instant of time.

Table 1 : Baseline values of the system. Parameters considered in the simulation as Healthy system

Parameter	Value
$x_0^{(1)} = x_0^{(2)}$	0 m
$\dot{x}_0^{(1)}$	0 m/s ²
$\dot{x}_0^{(2)}$	1 m/s ²
$k_1 = k_2 = k_3$	2000 N/m
k_n	10000 N/m
$c_1 = c_2 = c_3$	6 Nm/s
$m_1 = m_2$	5 kg

The 2-DoF system was modified by different levels of stiffness reduction which affect in the vibratory response of the whole system. The reduction of the stiffness was applied to k_n by 10%, 20% and 30%. The acceleration was recorded at any instant of time for each reduction of k_n . In this paper is demonstrated the sensitivity of the methodology and the capability to detected small changes into a dynamical system. The small changes induced by the reductions in k_1 and k_2 were easily detected, as it is shown in [11]. However, the changes in the k_n stiffness generated misclassification. This particular paper analyse the effect of nonlinear quadratic stiffness k_n on the vibratory response.

The acceleration was recorded from the free-decay response of the system by the integration of the Equation (8). The resolution of the signal was obtained using $N = 2048$ sampled points at a sampling interval of $\Delta t = 0.00125s$.

Ten realizations were obtained for each system by adding random white Gaussian noise at 20dB in: Without stiffness reduction (Healthy), 10% reduction of k_n , 20% reduction of k_n and 30% reduction of k_n .

SSA was then applied in the ten signals of the Healthy system with a window size of $W = 7$. The decomposition and their posterior reconstruction components were obtained to describe a new space where the data, from the different reductions of k_n , was projected following the methodology defined in §1.2 and §1.3.

3. RESULTS

SSA was applied in the system defined in §2. following the methodology detailed in §1. The eigen-decomposition of the original data based on the Healthy system is shown in the Figure 2. It is observed that after the fourth component the variance becomes steady and the information contained into these

components, in terms of variance, does not provide further information, hence they can be ignored. This judgement can be observed in the Figure 3(a) where the reconstructed signal was built by two components, and in Figure 3(b) where the reconstructed signal was built by four components. The use of more components increase the accuracy in the reconstruction of the signal.

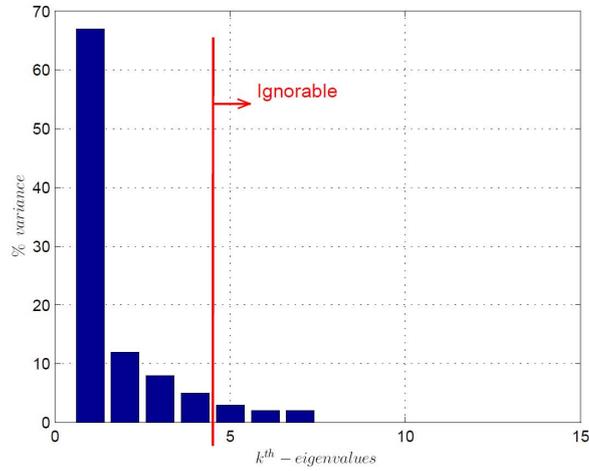


Figure 2 : Variance decomposition diagram based on the Healthy system. The graph shows the variance of only 15 components of 80.

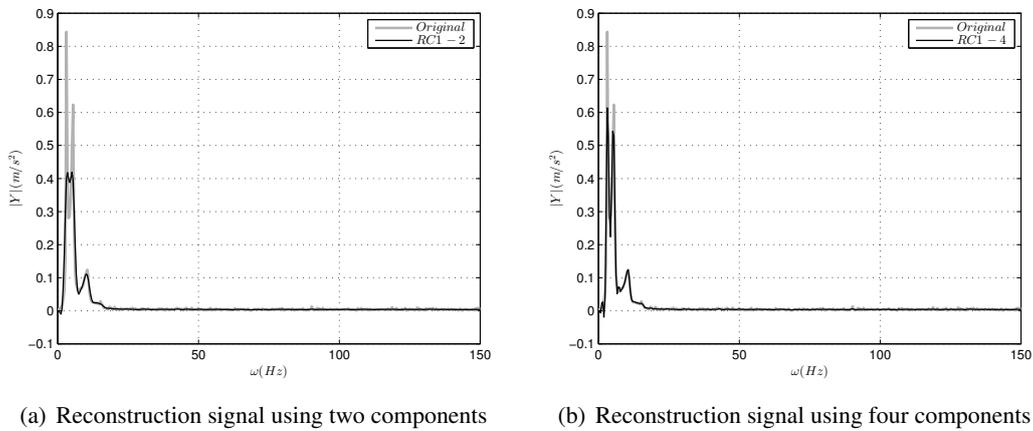


Figure 3 : Comparison between original and reconstructed signal for a Healthy system. The frequency scale has been limited to 150 Hz for visualization issues. The original scale reaches until 400 Hz.

The projection of the original signal onto the reconstructed components is traduced in a point onto the coordinates axes in the new space, as it is described in §1.2. A good cluster visualization was obtained by using only the two first components which contain 67% of variance. It can be observed in Figure 4.

The combination of SSA and the Mahalanobis distance demonstrates the improvement in the classification results. The classification is shown in the Figures 5(a), 5(b), 5(c) where is clearly demonstrated that by increasing the number of components, the classification is improved.

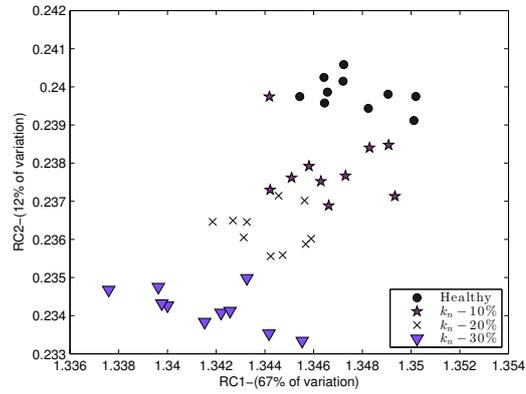
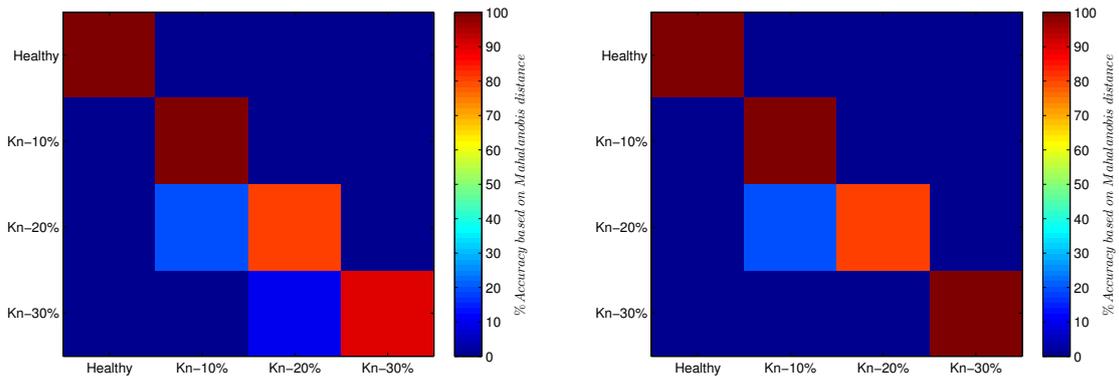
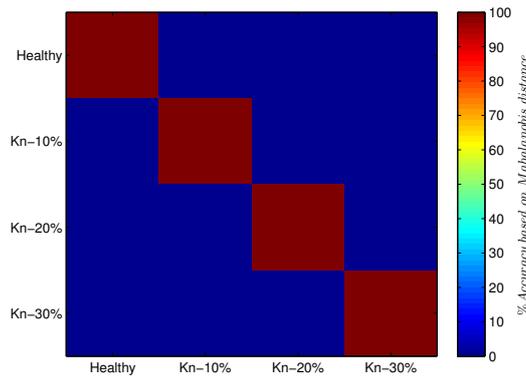


Figure 4 : Clustering effect in the reduction of k_n . Projection onto the two first Reconstructed components, RC1 and RC2.



(a) Considering the projection onto two RCs

(b) Considering the projection onto three RCs



(c) Considering the projection onto four RCs

Figure 5 : Confusion matrix of the Mahalanobis distances to each system.

4. CONCLUSIONS

An analytical study consisting of the reduction of stiffness in a 2-DoF system has been conducted to examine the effectiveness of SSA and Mahalanobis distance for damage detection and classification. The idea of this study is to apply SSA algorithm to the raw acceleration response for a better data compress analysis. The SSA algorithm was applied to reduce the dimension of the response of vibration and extract the only essential features. The data was then projected on the new space based on the two first reconstructed components of the healthy system, which contain the majority of the variance. The clustering effect was strongly demonstrated between different categories. Furthermore, the Mahalanobis distance was applied in the data-compressed obtained by SSA. The classification was improved using more reconstructed components. To conclude, one can envision that the methodology proposed can combine both the SSA data-compress algorithm and Mahalanobis distance towards damage detection and classification. The methodology guarantees the detection of a small changes in dynamical systems and it is proposed for detecting the non-linear changes in composite laminated beams by the behaviour of delamination.

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