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4 **Shockwaves in spillways with the particle finite**
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7 **element method.**
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22 **Abstract** Changes in direction and cross-section in supercritical hydraulic
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24 channels generate shockwaves which result in increase in flow depth with re-
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1 gard to that for uniform regime. These disturbances are propagated down-
2 stream and need to be considered in the design of the chute walls. In dam
3 spillways, where flow rates are often high, this phenomenon can have signif-
4 icant implications for the cost and complexity of the solution. It has been
5 traditionally analysed by means of reduced-scale experimental tests, as it has
6 a clear three-dimensional character and therefore cannot be approached with
7 two-dimensional numerical models. In this work, the ability of the particle
8 finite element method (PFEM) to reproduce this phenomenon is analysed.
9 PFEM has been successfully applied in previous works to problems involving
10 high irregularities in free-surface. First, simple test cases available in the tech-
11 nical bibliography were selected to be reproduced with PFEM. Subsequently,
12 the method was applied in two spillways of real dams. The results show that
13 PFEM is capable of capturing the shockwave fronts generated both in the
14 contractions and in the expansions that occur behind the spillway piers. This
15 suggests that the method may be useful as a complement to laboratory test
16 campaigns for the design and hydraulic analysis of dam spillways with complex
17 geometries.

18 **Keywords** Particle finite element method · Spillways · Shockwaves

19 **1 Introduction**

20 Shockwaves are frequently generated in dam spillways by uniform flow distur-
21 bances due to the presence of piers, curves or changes in the cross-section [34].

22 As a consequence, local maxima in flow depth are produced whose magnitude

1 can be much greater than the incoming uniform depth. This has important
2 practical implications in the dimensioning of the chute walls, which must be
3 designed with greater height to adequately convey the flow, with the obvious
4 consequences in terms of magnitude and cost.
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10 The design of the spillway must tend to the constant cross-section and slope
11 with the aim of approaching uniform flow and thus avoiding this phenomenon.
12 However, these geometric particularities are frequently unavoidable due to the
13 topographical and/or geological characteristics of the dam and spillway site.
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20 The hydrodynamic study of spillways has historically been carried out using
21 small-scale experimental tests. The computational methods available during
22 the period of greatest intensity in dam design and construction did not allow
23 these complex problems to be addressed, since they are not treatable with
24 two-dimensional models [22]. However, the numerical tools currently available
25 can be used for studying this type of 3-D problems with sufficient detail and
26 with an assumable computational cost.
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36 Numerical models are often used for the design and analysis of spillways
37 in combination with physical model tests [31], [30]. This allows the numerical
38 model to be validated for the geometry considered, so that the versatility of the
39 latter can subsequently be used to extend the catalogue of situations analysed
40 in alternatives studies. In addition, numerical models allow a more detailed
41 analysis of aspects that are difficult to measure in the laboratory, such as
42 stream lines or pressure in areas at risk of cavitation [32], [36].
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1 Numerical models have been primarily used to evaluate the discharge ca-
2 pacity of spillways [36], [26] and to optimize the approach conditions [31],
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4 thanks to the ease with which flow patterns can be analysed. The numerical
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6 schemes employed include eulerian [2], [26], and lagrangian approaches such
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8 as smoothed particle hydrodynamics (SPH) [29].
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12 However, the analysis of shockwaves is much less frequent. Some exam-
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14 ples with quasi-horizontal bottom have been addressed with depth-integrated
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16 approaches, which offered accurate results (e.g. [37], [33]). However, spillway
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18 chutes often feature steep slopes, which limits the applicability of 2D simplifi-
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20 cations. The particle finite element method (PFEM) is a lagrangian approach
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22 for the numerical modelling of fluid dynamics problems among many other
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24 applications [1]. The fact that the PFEM does not need a background mesh,
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26 and thus the user does not need to foresee what parts of the domain will be
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28 occupied by the fluid in later stages of the simulation, makes it especially ap-
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30 propriate to consider phenomena with strong changes in the free surface both
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32 in time and space [9], [8].
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36 The method has been employed to face a variety of problems in different
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38 fields of engineering, such as seakeeping [12], landslide-generated waves [25],
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40 [27], industrial forming processes [18], ground excavation [3], fluid-structure
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42 interaction [13], among others [15],[14], [7]. PFEM has also been previously
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44 applied to solving 3-dimensional free-surface flows, particularly in hydraulic
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46 structures [10]. It has also been employed to estimate air flow demand in
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48 bottom outlets [28], [20].
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1 This paper explores the possibilities of using PFEM in full-scale hydrody-
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3 namic analysis of spillways with geometric irregularities that generate shock-
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5 waves. Firstly, it has been applied to reference cases available in the classical
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7 literature including curves and channel contractions. Then, two examples of
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9 application to real dam spillways are presented, with different characteristics:
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11 the first features a mild slope and an abrupt convergence, reducing the use-
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13 ful width of the channel by more than 50 % in a length similar to the initial
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15 width. In the second case, the convergence is smoother, although it allows for
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17 a detailed analysis of the negative waves generated behind the piers.
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20 The article is completed with a brief reminder of the basis of the PFEM
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22 implementation used in this work and a discussion on the possibilities and
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24 limitations, as well as ideas for future work.
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28 **2 Numerical model**

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32 The PFEM is a documented method in the literature [19], well suited for con-
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34 tinua with large deformations and separation. In the PFEM, the domain is
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36 modelled using an Updated Lagrangian formulation [38]. That is, all variables
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38 are assumed to be known in the current configuration at time t . The new set of
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40 variables in the domain are sought for in the next or updated configuration at
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42 time $t + \Delta t$. The finite element method (FEM) is used to solve the equations
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44 of continuum mechanics. Hence a mesh discretising the domain must be gen-
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46 erated in order to solve the governing equations in the standard FEM fashion.
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49 When the mesh gets too distorted due to the large deformations it must be
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rebuilt to avoid inverted elements. The re-meshing step can be used to improve the mesh quality as well. The PFEM is therefore a compound method consisting in a combination of the Finite Element Method, an Updated Lagrangian formulation and a re-meshing algorithm.

The equations to be solved for a single phase fluid (like water) are the Navier-Stokes equations for incompressible fluids:

Momentum conservation

$$\rho \frac{Du_i}{Dt} = -\frac{\partial}{\partial x_i} p + \mu \frac{\partial}{\partial x_j} \left(\frac{\partial u_i}{\partial x_j} \right) + \rho f_i \quad (1)$$

for $i, j = x, y, z$

Mass conservation

$$\frac{\partial u_i}{\partial x_i} = 0 \quad (2)$$

for $i = x, y, z$

with

$$u = \bar{u} \quad (3)$$

for the wall nodes (considered as fixed fluid nodes) and

$$p = 0 \quad (4)$$

for the free surface fluid nodes.

In the equations above, the unknowns are p (pressure) and u (velocity). ρ and μ are the fluid density and dynamic viscosity, respectively, u_i are the velocities along the i th global (cartesian) axis, f_i are the volumetric accelerations

1 (only gravity acceleration is used in this work), and \bar{u} is the prescribed veloc-
2 ity. Note that fixing the three components of the velocity to zero is equivalent
3
4 ity. Note that fixing the three components of the velocity to zero is equivalent
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6 to assuming a no-slip boundary condition. This type of boundary condition
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8 was used in every fluid-wall interphase.
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10 The Lagrangian formulation implies that the nodes move with the fluid
11 velocity. In particular, the nodes located at the free surface follow the evolution
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13 of the boundary in a very accurate, natural way.
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15 According with the PFEM technique [19], equations 1 and 2 are discre-
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17 tised with a standard FEM mesh and then solved by means of an implicit,
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19 Fractional Step method. The elements used are simplex, 4-noded tetrahedra,
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21 with linear interpolation of both the pressure and the velocity. When the finite
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23 elements get very distorted, the mesh is re-generated, but the nodes and their
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25 information are conserved. Adaptive mesh refinement techniques can be used
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27 to improve the solution in zones where large deformations of the fluid occur.
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31 The details of the algorithm and our implementation was described in pre-
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33 vious publications [13], [16], [17]. In this paper, only the basic steps of the
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35 algorithm are succinctly described, together with some enhancements specifi-
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37 cally implemented for the present application.
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41 2.1 Basic steps of the PFEM

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44 In the PFEM, the mesh nodes in the fluid and solid domains are treated as
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46 particles that contain all the information of geometry, material and mechanical
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48 properties of the underlying subdomains.
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A typical solution with the PFEM involves the following steps.

1. The starting point at each time step is the cloud of points C in the fluid and the walls. The walls domain simply contains fluid nodes which have an externally imposed velocity and can be moved as a rigid body. For instance, ${}^n C$ denotes the cloud at time $t = {}^n t$ (Fig. 1).
2. The domain is discretised with a finite element mesh ${}^n M$ using the particles as the mesh nodes. We use an efficient mesh generation scheme based on the Delaunay tessellation [9].
3. The free surface is detected by means of the Alpha Shape Method [6], a geometrical criterion which removes the elements coming from the Delaunay Tessellation that are not considered as part of the fluid domain. The Alpha Shape Method compares the diameter of the circumsphere of each element with a reference, maximum allowed diameter. Bigger circumspheres correspond to either big elements (connecting distant nodes) or flat, distorted elements which can only be present in areas where the density of nodes drops considerably (in areas with a homogeneous density of nodes the Delaunay Tessellation generates high quality tetrahedra). Note that the free surface is not the border between two fluids, but the external boundary of the domain considered fluid.
4. The Lagrangian equations of motion for the overall continuum are solved using the standard FEM. The state variables in the next (updated) configuration for ${}^n t + \Delta t$ are computed: velocities, pressure, strain rate and viscous stresses.

5. The mesh nodes are moved to a new position ${}^{n+1}C$ where $n + 1$ denotes the time ${}^n t + \Delta t$, in terms of the time increment size.
6. Go back to step 1 and repeat the solution for the next time step to obtain a new ${}^{n+1}C$.

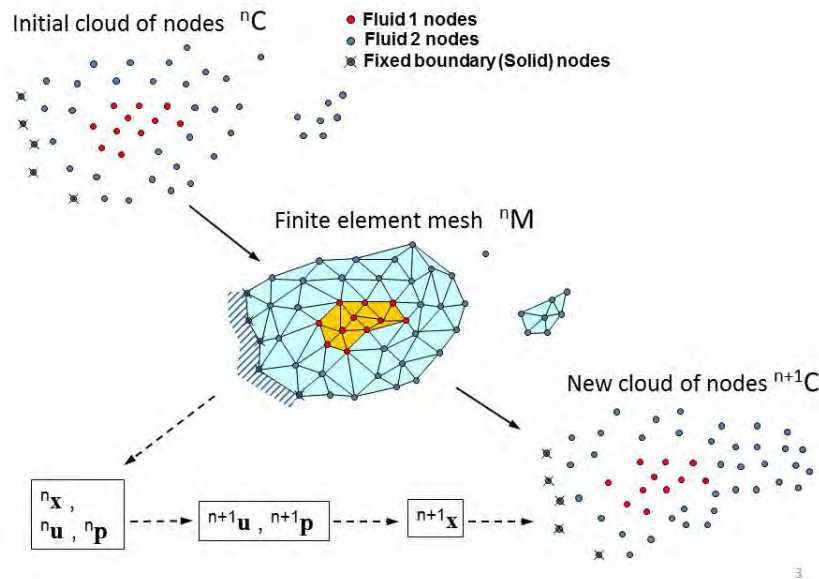


Fig. 1: Sequence of steps to update a “cloud” of particles (nodes) representing a domain containing two fluids and a solid boundary from ${}^n t$ to ${}^{n+1} t$.

2.2 Inlets

Fluid inlets are necessary when modelling spillways or channels as an upstream condition. Being the PFEM a fully Lagrangian method, the inlet condition

cannot just be modeled as a Dirichlet condition for the velocities because the nodes at the inlet surface get transported with the velocity field and move out of the surface where they were originally located. Thus, in the PFEM used for this work, inlets are conceived as successive layers of nodes that are inserted in the domain at a given location (usually a surface). Figure 2 depicts a scheme of the process in 2D. Every new layer of nodes must be considered exactly in the same way as a solid wall, with a Dirichlet velocity condition (step 2). For several time steps (step 3) the boundary condition is maintained until a certain distance from the inlet is reached and a new layer of nodes is inserted (step 4). This distance must be similar to the size of the elements. The old nodes are freed and the new ones are fixed to the same velocity as before. At this stage, both layers get connected by elements that are created automatically by the re-meshing process. After this, the newest nodes move with the imposed velocity (step 5) until they are replaced by a new layer of nodes in the same way it was done before (step 6).

3 Benchmark cases

Some classical cases have been collected to verify the ability of the PFEM to model irregularities in supercritical flow. They are described in this section. Only the essential information is included, since the experiments are well documented in the reference articles, which are conveniently cited.

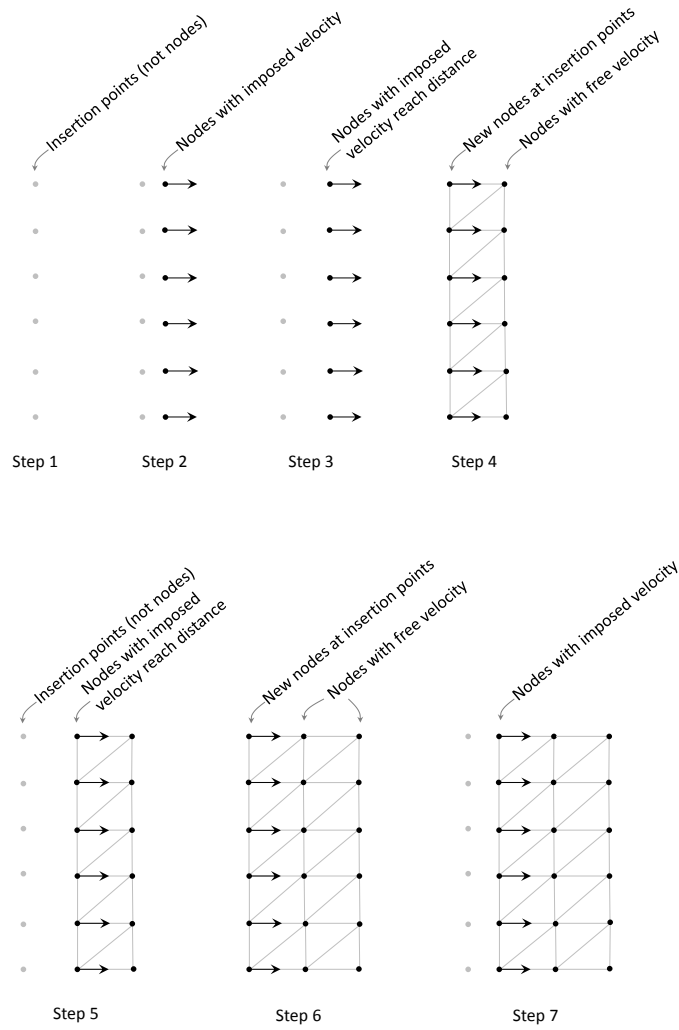


Fig. 2: Schematic process of node injection at an inlet

3.1 Chute bend

The change in the flow direction generated in a circular bend in supercritical regime results in two shockwaves: a positive one producing increase in flow

depth and a negative one with depth below uniform. Figure 3 shows a scheme of the phenomenon.

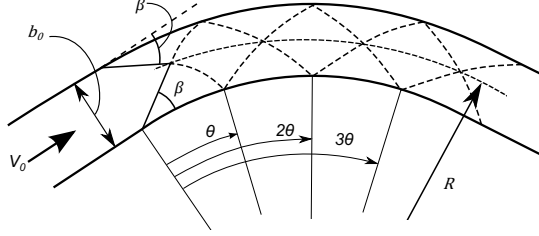


Fig. 3: Shockwave pattern in a rectangular channel bend with supercritical flow. Adapted from [5]

The angle β formed by both wave fronts with the initial flow direction depends on the Froude number of the incoming flow (F_0) and can be approximated by the expression ([5], p.452)

$$\sin(\beta) = \frac{1}{F_0} \quad (5)$$

In turn, θ can be computed as:

$$\tan\theta = \frac{\frac{b_0}{R}}{\left(1 + \frac{b_0}{2R}\right) \tan\beta_0} \quad (6)$$

Where b_0 stands for the channel width and R is the average radius of curvature. The maximum flow depth h_{max} can be estimated as:

$$h_{max} = h_0 F_0^2 \sin^2 \left(\frac{1}{F_0} + 0.5\theta \right) \quad (7)$$

Table 1: Bend in rectangular channel. Main features.

		Case B1	Case B2
Channel width	b_0	0.5 m	0.5 m
Average curve radius	R	3.25 m	4.25 m
Flow depth at inlet	h_0	0.05 m	0.05 m
Froude number at inlet	F_0	4	4
Inflow velocity	v_0	2.18 m/s	2.8 m/s
Mesh size		1 cm	1 cm
Number of elements		$2.19 \cdot 10^6$	$2.12 \cdot 10^6$
Theoretical maximum flow depth (Eq.7)	h_{max}	0.186 m	0.153 m
Theoretical location of maximum (Eq. 6)	θ	29.0°	23.3°

Two configurations have been modelled with the PFEM with parameters shown in Table 1, which also includes the theoretical values of the h_{max} and θ obtained with equations 6 and 7 .

Figure 4 shows the result of the numerical simulation, with both the location and the value of the first theoretical maximum flow depth. The exact location of h_{max} is not easy to identify in the numerical results, since the free surface at the outer wall is close to horizontal along a certain area. Nonetheless, the theoretical approximation falls within that area. The numerical result for h_{max} is 0.180 m and 0.159 m for $R = 3.25$ m and $R = 4.25$ m respectively, i.e. the deviation with respect to the theoretical maxima is 3.2% and 3.9%.

The results for the second and third maxima are less accurate (Table 2 and 3), especially for 2θ . This might be due to numerical diffusion, but also to the relatively high curvature, which can result in deviation from the theoretical

Table 2: Chute bend (Case B1). Flow depth maxima. Theoretical vs numerical results

Location (angle)	Flow depth (m)			
	Theoretical		Numerical	
	Inner wall	Outer wall	Inner wall	Outer wall
θ	0	0.186	0	0.180
2θ	0.186	0	0.042	0.094
3θ	0	0.186	0	0.144

Table 3: Chute bend (Case B2). Flow depth maxima. Theoretical vs numerical results

Location (angle)	Flow depth (m)			
	Theoretical		Numerical	
	Inner wall	Outer wall	Inner wall	Outer wall
θ	0.002	0.153	0	0.159
2θ	0.153	0.002	0.06	0.09
3θ	0.002	0.153	0	0.122

values [33]. Another possible source of deviation from the theoretical value is wave breaking [23], which indeed is observed in Figure 4.

3.2 Contraction

An abrupt contraction in a hydraulic channel in supercritical regime generates shockwaves resulting in increase in flow depth propagating downstream [34].

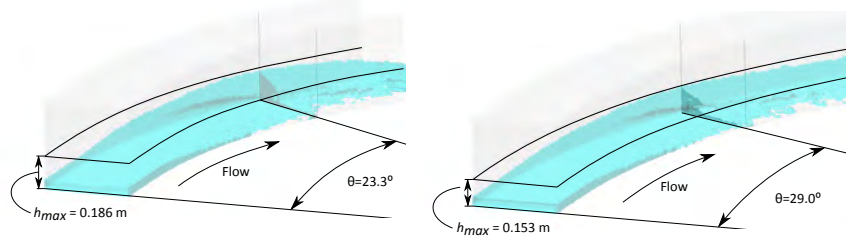


Fig. 4: Chute bend. The expected empirical values for h_{max} and θ are over-imposed on the numerical results. Left: Case B1; $R = 3.25m$. Right: Case B2; $R = 4.25m$.

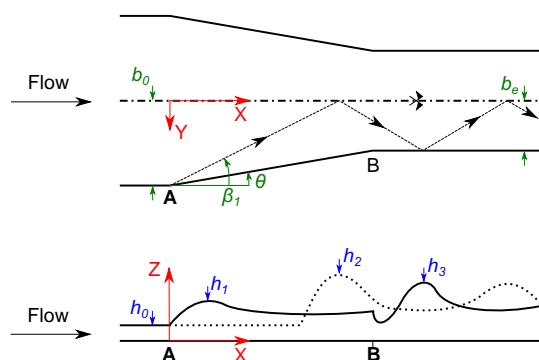


Fig. 5: Shock waves in a contraction. Local maxima in the axis (dotted line) and in the walls (solid line). Adapted from [24]

Reinauer and Hager [24] described the shape and location of the shockwaves both along the walls and the axis, as depicted in Figure 5.

The angle of the shock wave front can be estimated with the expression [21]:

$$\tan\theta = \frac{\tan\beta_1 \left(\sqrt{1 + 8F_0^2 \sin^2\beta_1} - 3 \right)}{2\tan\beta_1 + 8F_0^2 \sin^2\beta_1 - 1} \quad (8)$$

Where: θ is the angle formed by the chute wall and the direction of the incoming flow, β_0 is the shockwave angle and F_0 is the Froude number before the contraction.

Reinauer and Hager [24] proposed expressions to calculate the maximum flow depth on the walls and axis, as well as the location of the first maximum:

$$X_1 = (2 + 0.126\alpha)\theta + (1.4 - 0.015\alpha) \quad (9)$$

$$h_1 = h_0 \left(1 + \frac{1}{\sqrt{2}}S_0\right)^2 \quad (10)$$

$$h_2 = h_0 \left(1 + \sqrt{2}S_0\right)^2 \quad (11)$$

$$h_3 = h_0 \left(w^{-1} + 1.86S_0^{0.5} - 0.2\alpha^{0.6}\right) \quad (12)$$

$$w = \frac{b_e}{b_0} \quad (13)$$

$$S_0 = \theta F_0 \quad (14)$$

Where X_1 is the location of h_1 , $h_{1..3}$ is the value of the three first maxima, alternatively located on the wall and the axis. α is the channel slope. θ is the angle formed by the wall with the initial flow direction. h_0 , F_0 and b_0 are the flow depth, the Froude number and the channel width before the contraction, respectively. b_e is the channel width downstream of the contraction.

Table 4: Parameters for the contractions analysed

		Case C1	Case C2
Channel slope	θ (rad)	0.186	0.169
Angle of the chute wall	R	3.25 m	4.25 m
Channel width at inlet/outlet	b_0 (m)	0.5/0.3	0.5/0.15
Flow depth at inlet	h_0	0.05 m	0.05 m
Froude number at inlet	F_0	6	6
Inflow velocity	v_0	4.2 m/s	4.2 m/s
Mesh size		1.2 cm	1.2 cm
Number of elements		$1.29 \cdot 10^6$	$1.11 \cdot 10^6$

Two of the settings available in the literature have been selected to be modelled with the PFEM to assess its capability to reproduce shockwaves in these configurations, with different conditions. The parameters defining both tests are shown in Table 4.

The maximum mesh size in both cases was set to 12 mm. This value ensured having at least 3 elements in depth in the whole domain. Also, only half of the original geometry was considered, taking advantage of the symmetry of the design.

3.2.1 Contraction case C1

Figure 6 shows a general view of the numerical result, where the flow depth maxima can be identified. They are alternatively located on the wall and axis, as expected. The theoretical values of X_1 , h_1 , h_2 and h_3 have been computed with Equations 9, 10, 11 and 12. A direct comparison with the numerical results

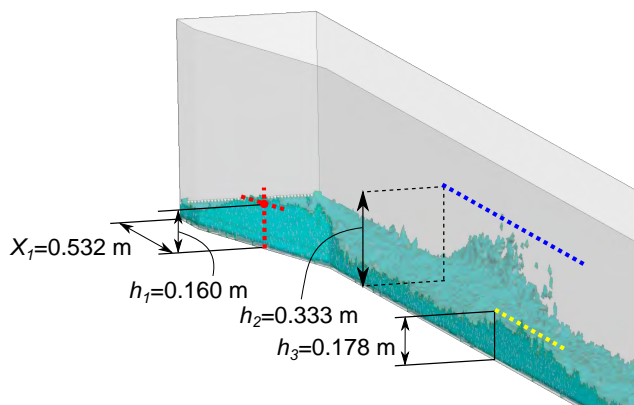


Fig. 6: Contraction case C1. The expected empirical values for X_1 , h_1 , h_2 and h_3 were placed over a view of the numerical result.

cannot be made, since the intrinsic properties of PFEM result in irregular shape of the free surface, with some elements separating from the main fluid body. Therefore, planes –horizontal for h_1 , h_2 y h_3 and vertical for X_1 – have been drawn and superimposed on the numerical results for comparison. It can be seen that all results lie within the correspondent location.

The plan view of the numerical result (Figure 7) allows approximating the resulting angle β_1 formed by the shockwave front, which in this case presents some curvature. The estimated direction (18°) is 5% lower than the empirical estimate computed with Equation 8 (19.2°).

3.2.2 Contraction case C2

In this case, the results of the PFEM were compared to the experimental measurements published by Reinauer [21]. For that purpose, the profiles of the

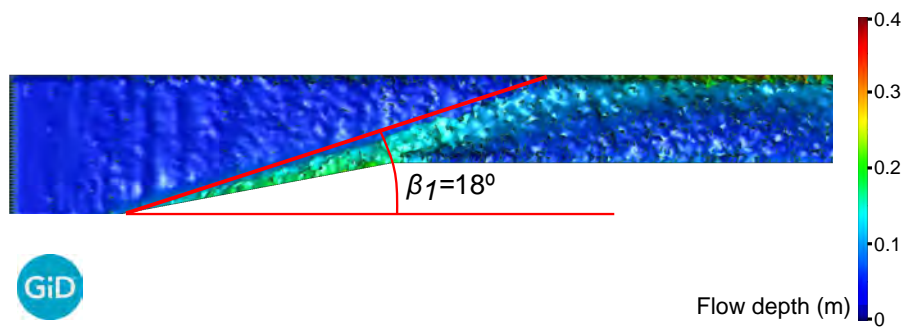


Fig. 7: Contraction C1. Numerical results (plan view). The shockwave front features a small curvature. Its direction can be estimated as forming 18° with the main flow direction. This is 5% lower than the empirical estimate computed with Equation 8 (19.2°).

free surface along the chute wall and axis were extracted and superimposed over the experimental ones in Figure 8. The results coincide qualitatively in terms of the alternation of maxima between the axis and the wall, including the local minimum in the wall immediately downstream of the contraction. The main discrepancy is observed in the second maximum on the axis, which moves downstream in the numerical model.

As in the previous case, the angle of the shockwave front has been approximated and compared to the empirical value given by Equation 8. The result is shown in Figure 9.

The difference in β_1 is coherent with the location of the flow depth maxima in the numerical model, slightly downstream as compared to the empirical estimate (Figure 8).

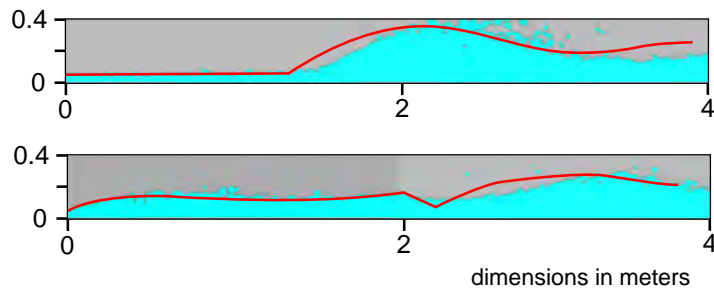


Fig. 8: Contraction case C2. Side view of the numerical results as compared with the flow depth measured in the experiment (red lines) [21]. Side view.

Top: Chute wall. Bottom: Channel axis

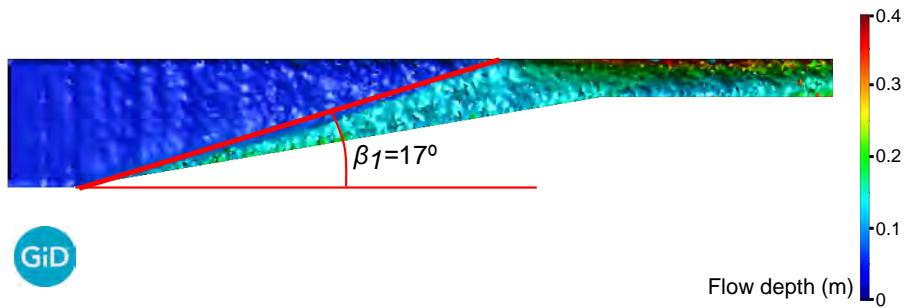


Fig. 9: Contraction case C2. Numerical results (plan view). The direction of the shockwave front can be estimated as forming 17° with the main flow direction.

As in the previous case, the empirical estimate given by Equation 8 (18.2°) is 5% higher.

4 Real scale tests

In this section, two cases are presented in which the PFEM is applied to analyse the hydraulic behaviour of spillways with different degree of convergence. Both share geometric features resulting in the formation of shockwaves.

4.1 Case study #1

4.1.1 Model description

The first spillway analysed includes a chute with a mild slope (11.1 %). It features four spans 13 m wide regulated by gates supported by three piers: the central pier is 3.5 m wide and has a rectangular profile at the downstream end, and the other two are 2 m wide, with downstream hydrodynamic profile and shorter length. The total width of the initial section is 59.5 m and after a transition along 55 m it joins the discharge channel, which has a constant useful width of 21 m. In this case, the side channels feature sloping bottom to direct the flow towards the axis. In addition, they are slightly elevated with respect to the central channel, which in turn is aligned with that of the downstream chute. Given that the focus of this study is on the shockwaves generated by the change in cross-section, the energy dissipation structure was not analysed. Figure 10 shows the geometry and the main dimensions of the model.

A discharge rate of $1,800 \text{ m}^3/\text{s}$ has been considered. This value corresponds to one of the experimental tests for which data are available (the actual flow in the laboratory was obtained by applying hydraulic similarity to the aforementioned $1,800 \text{ m}^3/\text{s}$). The gates remain fully open while evacuating this flow, therefore the weir operates in free flow regime. This situation has been reproduced in the numerical model by applying an inlet boundary condition corresponding to the critical regime (flow depth = 4.98m ; inflow velocity

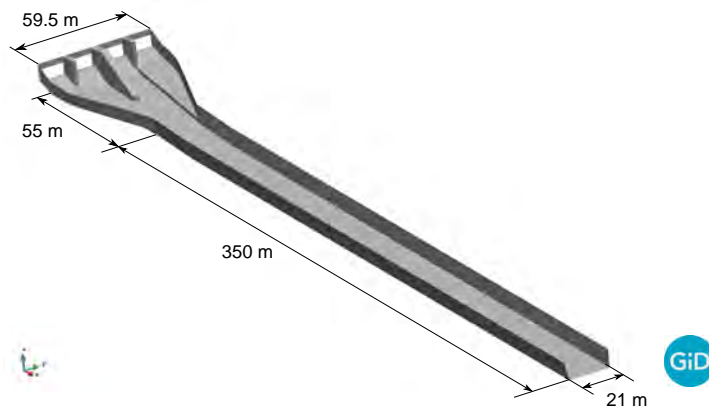


Fig. 10: Geometry of the numerical model for Case study #1

= 6.95m/s), as described in Section 2.2. A constant mesh size of 0.5 m has been selected in accordance with the expected flow depth, resulting in about three million elements once the flow is stabilised. These magnitudes constitute a compromise between a sufficient level of detail to reproduce the flow depth variations with a file size that allows for efficient post-process, and with moderate computation time.

4.1.2 Expected behaviour

In addition to the convergence of the whole section, which goes from 59 to 21 m as mentioned above, the central channel also has a convergence so that its useful width goes from 26 m at the inlet (two bays) to 21 m of the chute. As there is also a step between the bottom of this area and that of the outer spans, additional shock waves are expected to occur. Both convergences are indicated in Figure 11.

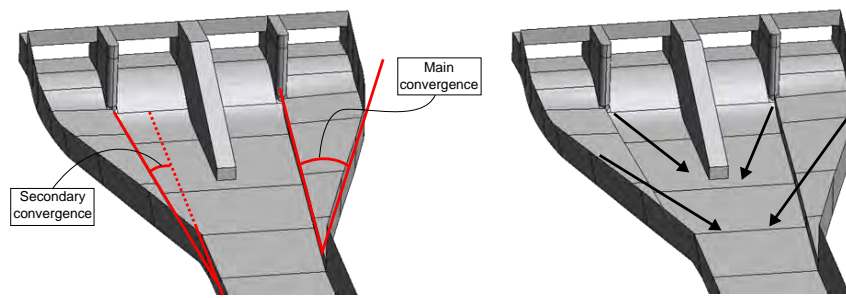


Fig. 11: Left: Convergence of the chute in Case study #1. Right: Expected shockwaves

It is well known that contractions of this type with supercritical flows generate shockwaves propagating downstream, similar to those described in Section 3.2. The angle β_1 formed by the wave front can be obtained with Equation 8. According to this theoretical formulation, two pairs of wave fronts should appear, corresponding to the two width changes shown in Figure 11, which will propagate downstream.

4.1.3 Numerical results

The numerical modelling performed with PFEM has been capable to reproduce the formation of shockwaves observed in the physical model and described by Reinauer and Hager [24], both in the walls and in the spillway axis, due to the confluence of the shock waves (Figure 12).

The complex geometry of this spillway prevents local maxima from being estimated using the formulas mentioned above. As an example, h_2 in this case results from the combination of the increase in flow depth produced by the

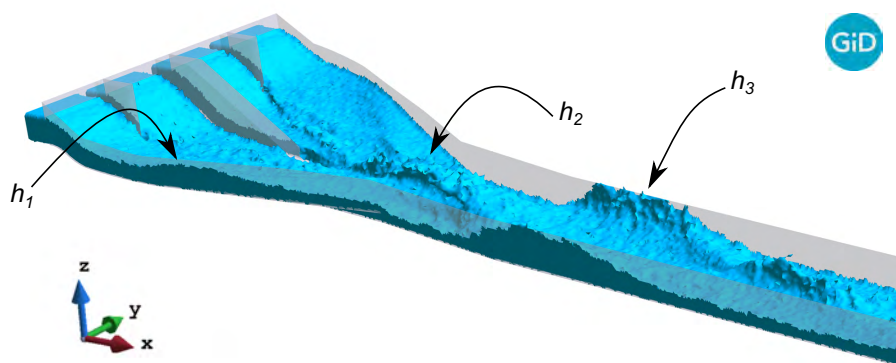


Fig. 12: Local maxima in the chute alternates between the walls and the axis.

convergence and that due to the expansion behind the central pier, known as *roostertail* [34], [35].

The numerical results have been compared to those obtained in the laboratory. Figures 13 and 14 include different views of the experiments and the corresponding numerical results. In the latter, the magnitude of the velocity component perpendicular (along the Y axis in the reference system used) to the main flow direction (X direction) is plotted for better visualisation of the shockwaves. This and other variables, which require complex instrumentation to be measured in laboratory, can be easily analysed in the numerical models.

The results presented for this case study show PFEM's ability to qualitatively capture the phenomenon. For a quantitative evaluation, they have been compared to the theoretical values of the angle formed by the wave front, and to the flow depth measured in the laboratory with the considered flow rate.

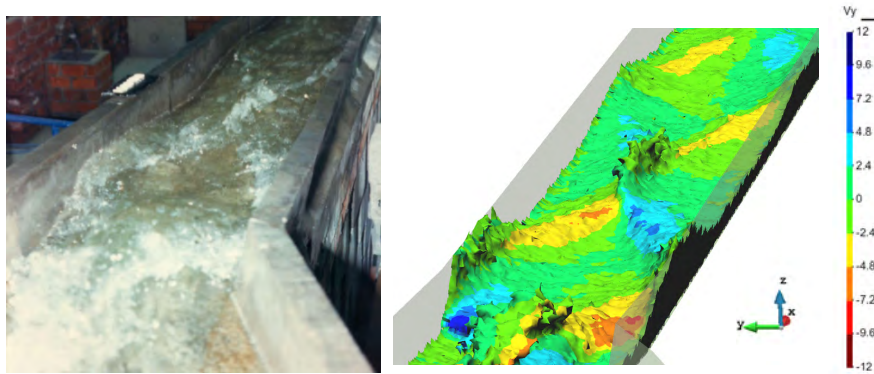


Fig. 13: Comparison between numerical and experimental results for Case Study #1. View from upstream.

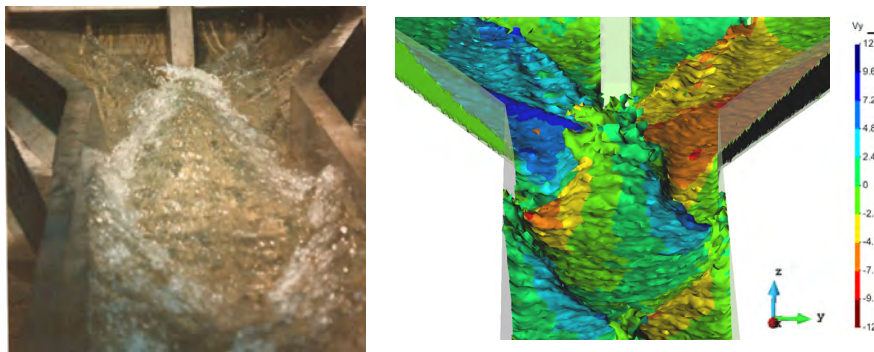


Fig. 14: Comparison between numerical and experimental results for Case Study #1. View from downstream.

The theoretical direction of the wave front β_1 can be obtained with Equation 8 by entering the value of the incoming Froude number and the angle formed with the main direction of the channel. With the features of this case study, a β_1 value of 27.2° is obtained for the secondary convergence. In Fig-

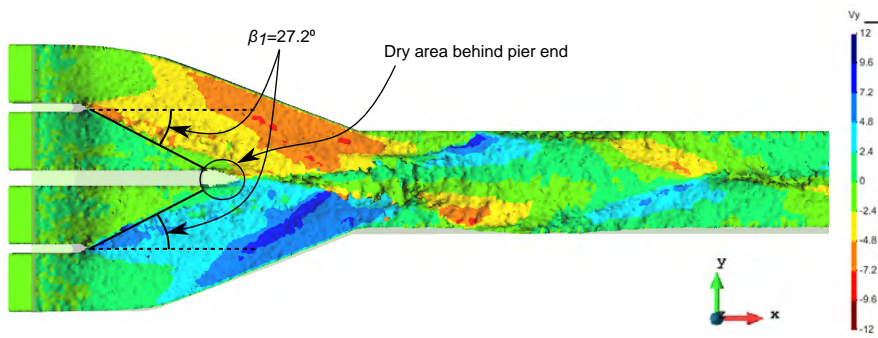


Fig. 15: Comparison between the empirical estimate for the shockwave front angle β_1 and the numerical results. The magnitude of the velocity component perpendicular to the main flow direction is plotted for better visualisation of the shockwave location.

Figure 15 this direction has been superimposed on a detailed plan view of the numerical results.

Finally, the flow depth along the right chute wall has been compared to that registered in the laboratory. Figure 16 shows that the first two maxima are coincident in magnitude, although they are located slightly downstream in the numerical model. This behaviour is similar to that observed in section 3.2.2.

4.2 Case study #2

The second real spillway modelled corresponds to a double curvature arch dam, with a height of 102 m. The spillway is located on the left abutment, with 3 gate-controlled bays of 6.0 m separated by 2.0-m-wide piers. The chute

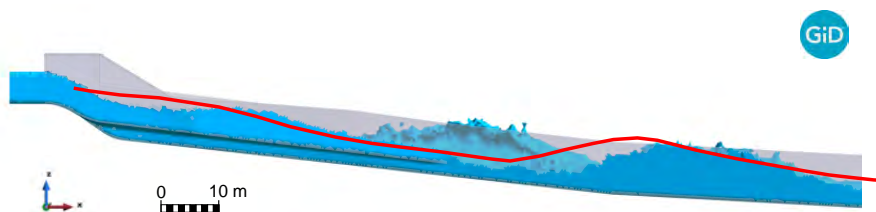


Fig. 16: Depth along the right chute wall. Comparison between the experimental results (red line) and the numerical model.

Table 5: Boundary conditions and mesh properties for Case study #2

Flow (m^3/s)	Flow depth at inlet (m)	Inflow velocity (m/s)	Mesh size (m)	# Elements
160	2	4.44	0.25	$1.48 \cdot 10^6$
450	4	6.25	0.3	$1.41 \cdot 10^6$

walls feature a mild convergence, with a total channel width changing from 22.0 m to 12.0 m at the entrance of the sky jump, after 90 m with a 45 % slope.

The spillway has been reported to perform satisfactorily, as can be expected from such a gentle convergence. Nonetheless, piers typically generate local maxima termed roostertails, which were the main goal of this case to be modelled with PFEM. Two scenarios were considered, with discharge flow of 160 and 450 m^3/s . The upstream boundary condition was set to match with the critical flow under the gates, assuming free flow (Table 5).

Since no experimental results are available for this specific geometry, we compared the numerical results with empirical expressions to estimate the magnitude of the maximum flow depth behind the piers. Reinauer and Hager

[22] described their location and shape for channels with horizontal bottom and Froude number above 5. For piers with rectangular end, they described a dry area limited by the flow from both sides of the pier which then collide and generate a local maximum. The waves propagate downstream and form a second local maximum once they reach the chute wall or another wave. These features can be observed in the numerical model.

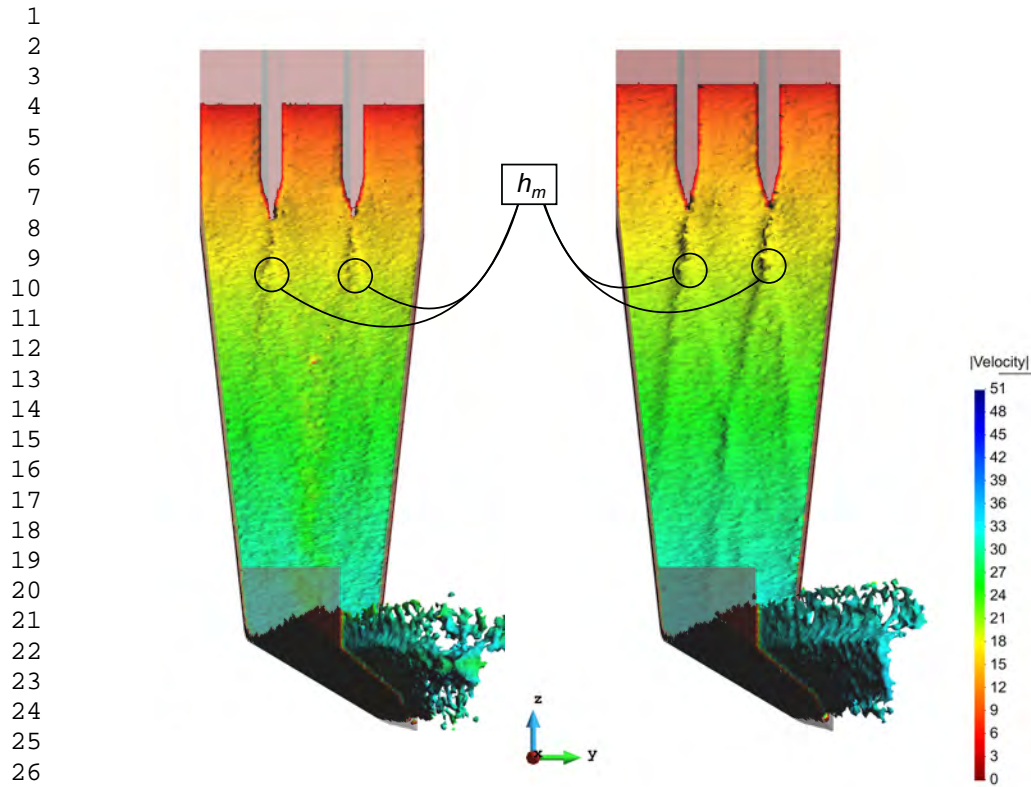
The PFEM results show the roostertails behind the piers (Figure 17). As expected, the magnitude of the local maxima increased with higher flow rate. Figure 18 includes a close-up view.

The results show that the mild convergence in the chute width does not generate relevant shock waves.

The magnitude of the local flow depth maxima can be compared to the result of experimental formulas. The above mentioned study of Reinauer and Hager has been recently extended by Xue *et al.* [35] with additional experimental data including sloping bottom i . These authors propose the following expression to compute the maximum flow depth h_m for piers with elliptical end of semi-major axis a and $h_0/b \leq 1.5$, where h_0 is the flow depth at the tail part of the pier and b is the pier width:

$$h_m/h_0 = 2.5(h_0/b)^{0.5} \exp(-0.74 \cdot a/h_0 - 1.11 \cdot i) \quad (15)$$

This expression cannot be directly applied to this case study, since the geometry of the pier end is not elliptical. Therefore, we verified that our numerical results are intermediate between those obtained for rectangular and



29 Fig. 17: Numerical results for $160 \text{ m}^3/\text{s}$ (left) and $450 \text{ m}^3/\text{s}$ (right). The loca-
30 tion of the flow depth maxima is highlighted.
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34 elliptical pier ends with equivalent dimensions (Figure 19). Table 6 shows the
35 results.
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38 5 Discussion

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45 The results obtained in the test cases fit qualitatively with what was observed
46 in the experiments. Similarly, the key magnitudes (maximum flow depth and
47 its location) have been adequately reproduced with PFEM. Comparison with
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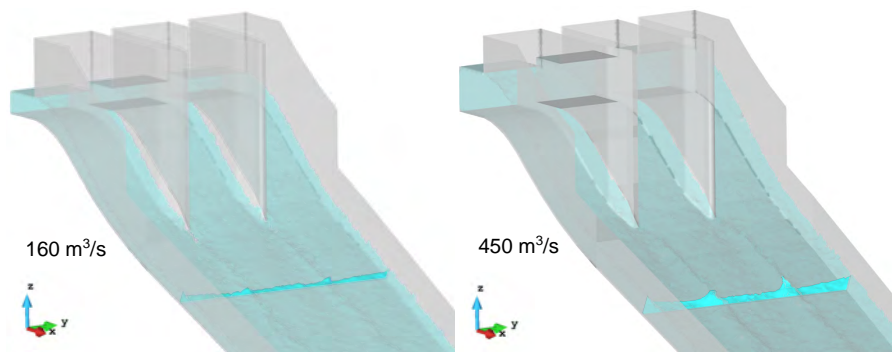


Fig. 18: Detail of the upstream reach of the chute. The cut highlights the flow depth maxima due to the roostertails.

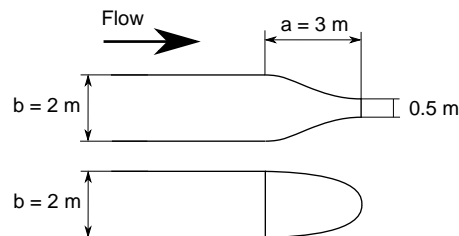


Fig. 19: Geometry of the piers in Case Study #2 (top) and that for the equivalent semi-elliptical and rectangular (bottom).

Table 6: Comparison of experimental and empirical results for the flow depth maximum behind piers with $b = 2m$ and $i = 0.469$

		Rectangular		Case study #2	Elliptical
		Eq 15; $a = 0m$		Numerical	Eq 15; $a = 3m$
Q (m^3/s)	h_0 (m)	h_0/b	h_m/h_0	h_m/h_0	h_m/h_0
160	1.05	0.48	1.024	0.704	0.1
450	2.3	1.04	1.52	0.786	0.52

1 experimental cases should be analysed with caution as the information avail-
2 able from the tests is limited. It has already been mentioned that in some
3 cases it is not easy to determine precisely the maximum flow depth, since ei-
4 ther a relatively large area with similar depth is observed (Figure 4), or strong
5 irregularities appear on the free surface (Figure 6).
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12 In the cases analysed corresponding to contractions in rectangular channels,
13 the angle formed by the shock wave front is slightly lower than that obtained
14 from Equation 8. This causes the maximum depth to shift downstream in the
15 numerical model, relative to the empirical estimate. Although in some cases
16 it is not easy to determine the exact position of these maxima, this effect is
17 clearly observed in the Figure 8.
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25 The aforementioned shockwave front in contractions presents a slight cur-
26 vature which has also been observed by Reinauer in laboratory [21].
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30 The results of the Case Study #1 offer a good approximation to the flow
31 pattern observed in the laboratory. The quantitative comparison is limited
32 because the experimental data correspond to some old tests for which some
33 details are unknown such as the uncertainty in the depth measurements.
34 Therefore, we consider that the most valuable result is the visual compari-
35 son with representative images (Figures 13 and 14). However, the numerical-
36 experimental comparison of the flow depth along the right chute wall (Figure
37 16) gives similar results to those obtained for test cases: The numerical model
38 offers a good approximation to the maximum depth, which are displaced down-
39 stream with respect to the experimental values. In this same Case Study #1
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1 it is important to emphasize that the angle of the shockwave front fits with
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3 the theoretical estimate (Figure 15).
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5 The Case Study #2 has major limitations regarding quantitative evaluation
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7 of the results, as there are no experimental data to compare with. Furthermore,
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9 the geometry at the end of the piers does not correspond to those for which
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11 empirical formulas are available (rectangular, semi-elliptical). In spite of that,
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13 the results fit qualitatively with the observed behaviour, and the local depth
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15 maxima due to the roostertails behind the piles are observed, with magnitudes
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17 within the expected ranges taking into account the specific geometry.
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22 **6 Summary and conclusions**

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25 The possibilities of PFEM to model shock waves in hydraulic structures with
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27 supercritical regime have been shown in different configurations: laboratory
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29 tests corresponding to curves and convergence have been reproduced, and
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31 subsequently 2 spillways of real dams featuring different geometries have been
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33 modelled. The results show that PFEM is a useful tool for numerical modelling
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35 of this type of phenomena, offering a good approximation with an assumable
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37 computational cost, particularly for the first shock wave.
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40 The main limitation of the presented approach is the lack of consideration
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42 of the interaction with air, which causes increase in flow depth in a supercrit-
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44 ical regime [4]. In previous works on hydraulic assessment of bottom outlets,
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46 interaction between water and air was considered and meaningful results were
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48 obtained [28], [20]. However, consideration of air entrainment in spillways im-
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1 plies using a large domain of analysis to model the surrounding air, together
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3 with a dense mesh in the interface between both fluids, to account for their
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5 interaction. This results in excessive computational cost with the current ver-
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7 sion of the code. The authors are currently working in optimizing this PFEM
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9 implementation to consider aeration. The mentioned works on bottom outlets
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11 and other precedents [12] are the basis of the new developments.
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14 The same problem arises in many laboratory tests, where aeration cannot
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16 be adequately modelled due to scale effects [11]. In spite of that, laboratory
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18 tests have been extensively used in the design and evaluation of dam spillways.
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20 Therefore, the model presented can be useful as a complement to experimental
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22 tests. The application of PFEM in these settings could reduce costs, enhance
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24 the analysis by considering flow patterns and stream lines and, above all,
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26 enable a wider spectrum of possible solutions to be considered, thanks to the
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28 greater flexibility of numerical models for the generation of geometries that
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30 can be laborious to build in the laboratory.
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34 **Acknowledgements** The research was supported by the Spanish Ministry of Economy
35
36 and Competitiveness (Ministerio de Economía y Competitividad, MINECO) through the
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38 project CALA (RTC-2016-4581-5).
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43 **7 Conflict of interest**

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46 On behalf of all authors, the corresponding author states that there is no
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48 conflict of interest.
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