



On the Optimisation of Road Vehicle Leading Edge Radius in Varying Levels of Freestream Turbulence

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ABSTRACT

It has been recognised that the ideal flow conditions that exist in the modern automotive wind tunnel do not accurately simulate the environment experienced by vehicles on the road. This paper investigates the effect of varying one flow parameter, freestream turbulence, and a single shape parameter, leading edge radius, on aerodynamic drag. The tests were carried out at model scale in the Loughborough University Wind Tunnel, using a very simple 2-box shape, and in the MIRA Full Scale Wind Tunnel using the MIRA squareback Reference Car. Turbulence intensities up to 5% were generated by grids and had a strong effect on transcritical Reynolds number and Reynolds sensitivity at both model scale and full scale. There was a good correlation between the results in both tunnels.

INTRODUCTION

It has always been the case that wind tunnel testing is only an approximation of the real world. Over recent decades wind tunnel engineers have added moving ground planes, boundary layer suction, adaptive walls, and other improvements to try to bring the wind tunnel model closer to the aerodynamic environment of the real car. Recently, more attention has been paid to modelling the turbulence environment of the real world, in the tunnel.

Real world ambient wind data are routinely collected for use by civil engineers, however much of this concentrates on wind velocities and turbulence at or above 10m. There are some data in the ESDU database [1] for as low as 3m, that show some interesting information, but it is not legitimate to simply extrapolate these data down to ground level. It is shown that increasing the roughness increases the turbulence intensity at 3m from approximately 11% for the lowest roughness (snow covered farmland, flat arid desert) to more than 36% for the highest (city centres, forests) and trends are drawn of turbulence intensity and length

scale. However, the length scale decreases with increasing roughness, from a maximum of 100m down to just over 1m at the highest roughness.

Richards et al. [2] investigated atmospheric turbulence closer to the ground. Their research was also aimed at structural engineering, but they were interested in the "wind loads on a variety of walls, fences and buildings". Measurements were taken for various heights from 0.115m to 10m using 3-component ultrasonic anemometers. Spectra and coherence information was published in order to compare data sampled at different frequencies and offer spectral models for the atmosphere at low levels. The results show that the turbulence spectra do vary significantly between 10m and 1m. There is some small increase in the spectra for u and v components at frequencies between 0.01Hz and 0.1Hz, and a larger increase between 0.1Hz and 10Hz. It is the authors' understanding that such an increase in high frequency energy indicates a transfer from large scale to smaller scale turbulence (from various general sources on turbulence, eg. Bradshaw [3]). The w component data, however show that the spectra for vertical turbulence are significantly reduced in energy for frequencies between 0.001Hz and 1Hz, and increased between 1Hz and 10Hz.

Several studies have been carried out to examine the turbulence experienced by moving road vehicles, including by Watkins and Saunders [4, 5]. They made measurements from moving and stationary vehicles on open roads with few obstructions, and roads with significant housing or trees by the roadside, and found that typical longitudinal turbulence intensities varied from 2.5% to 5%, and lateral intensities from 2% to 10%. In both cases the surrounding countryside was open and flat.

Modelling the real world in a wind tunnel is difficult (Watkins and Saunders [5]). The larger length scales experienced in the real world, that can be said to have a defined effect on road vehicles (ie. micrometeorological phenomena such as gusts and wind turbulence) tend to

be of the same order as the vehicle size, or bigger. Close to the ground, the atmospheric boundary layer has a sheared velocity profile. This means that for a vehicle moving at constant speed in a crosswind, there will be a variation in the total resultant wind speed and direction between ground level, where the ambient wind speed is at its lowest, and the roof of the vehicle. Passive grid turbulence therefore does not accurately represent the complete real world.

However, there is some evidence for dividing the micrometeorological scale further. Bearman [6] states in his review of the effects of turbulence on bluff bodies that if the length scale is large, in comparison to the body, then it is seen as a correlated unsteady mean flow of varying amplitude and dimension. If it is small, it will tend to decay rapidly over the length of the body, although it will also be distorted by the streamline curvature around the body. Smaller length scales may affect the mean flow field around the body, and therefore the mean body forces such as lift and drag.

It may therefore be that although grid generated turbulence is not completely representative of the real world, it still has a part to play in the development of road vehicles, as a device to introduce the smaller scale turbulence that is present in the real world in some form.

Some study has been made of comparisons between wind tunnel data using grids and real world drag reduction. Watkins [7] looked at measurements made of the drag reduction from using a spoiler on a lorry cab in the wind tunnel and compared them to results gained from on road testing. Although Watkins showed that the two sets of results did not match, the method used to measure the drag of the vehicles on the road gave a significant scatter, making the results difficult to interpret.

Wiedemann [8] looked at the effect of turbulence on the "effective Reynolds number" for a simple Reynolds sweep experiment on a car-like body in an open jet wind tunnel. The results gave the drag of the vehicle as being lower with turbulence, although, in the clean tunnel, the drag coefficient (C_D) did not become stable with Reynolds number at high speed. If higher Reynolds numbers were used it is possible from the trend that the drag coefficient with turbulence would cross over the clean C_D and become higher.

In Cogotti's [9, 10] tests at full scale, the C_D was higher, with the turbulence generating "wings" in use, than for the clean tunnel measurements. However, because the wings introduced a velocity profile, it is stated that it was difficult to identify a reference velocity. Several options are given in the original paper but the point is left open for discussion. In the later work, the question of reference velocity is left unanswered for the case of testing the difference caused by only adding turbulence. The focus is on the simulation of turbulence due to ambient wind, and aiming to simulate this by using a different speed for the rolling road and for the tunnel speed, and using the rolling road speed as a reference.

It seems that, in this case, most of the increase in drag is due to the increased dynamic pressure, rather than the turbulence alone. This seems an accurate depiction of the real world, at least at 0° yaw, and perhaps it would be possible to optimise a car under this particular condition, although it would be difficult to compare to any other type of test. If the question is the effect of turbulence alone, the experience from this work is that the accuracy of the reference velocity is of significant importance for comparing results at different turbulence levels.

A study was also published by Newnham [11], examining the effect of grid-generated turbulence on front edge radius optimisation. A very simple model was used, designed to have the same width to length to height ratios as a typical 4x4 vehicle. All the normal detail was removed to give a 1-box shape with a flat floor and roof, flat sides, and a flat rear face. A selection of front sections for varying the leading edge radii on the top and side edges of the front face were constructed. Reynolds sweeps were conducted in the clean tunnel, and comparison made with Cooper [12]. The experiment was then extended by using two different turbulence grids, changing the turbulence intensity in the tunnel from approximately 0.2% to 1.4% and 1.5%. It was shown that the increase in turbulence modified the behaviour of the flow over the model, giving a different characteristic curve for the drag coefficient against Reynolds number than in the clean tunnel. It was suggested that the increased turbulence intensity, rather than simply increasing the effective Reynolds number, increases the effective edge radius. The work also discussed the effect of the turbulence on the final drag coefficient of the model. In the original testing, additional turbulence increased the drag coefficient, but in an unpredictable way, with the first, lower turbulence grid producing the highest C_D values.

This paper presents an investigation of road vehicle edge radius optimisation techniques in turbulent flows, giving a comparison of results at model scale and full scale, using turbulence generating grids to raise the freestream turbulence intensity in the wind tunnels. At model scale, ten radii are tested in five turbulence levels, and at full scale, seven radii are tested in three turbulence levels.

EXPERIMENTAL METHOD

MODEL SCALE TUNNEL

A basic carlike model was developed, based on the aspect ratio of a typical offroad vehicle using the same approach of Newnham [11], but designed to be a basic 2 box shape, with replaceable front sections and a square back. Each front section had a unique radius around the top and side edges, and all were flat on the bottom edge. Following the previous work by Newnham, the main model section had 70mm radii on the A-pillars and roof to windscreen edge, which was expected, from the results of the 1-box shape, to ensure that the flow did

not separate significantly in these areas. The aim was to produce a measurable separation and reattachment effect for the front section and its interaction with the bonnet and windscreen only, and to minimise the chances of separation on the rest of the body. The model is shown in *Figure 1*. The front section radii varied from 40mm to 100mm in 10mm steps.



Figure 1 2-box model installed in the Loughborough tunnel

The Loughborough 1.3m * 1.9m tunnel was used. The tunnel is an open circuit, closed jet type with a maximum speed of 45m/s with a model installed. Reynolds number sweeps were conducted for each front radius in each turbulence condition by varying the wind tunnel speed between 5m/s and 45m/s, in 2.5m/s steps. This gave a range of Reynolds numbers based on the square root of frontal area (Re_A) from $1.3 \cdot 10^5$ to $1.2 \cdot 10^6$. For the higher turbulence grids, the maximum Reynolds number was reduced by the additional blockage, to $1.0 \cdot 10^6$ for the 3.5% grid and $8.3 \cdot 10^5$ for the 5.1% grid, because of the increased power requirement. This was the main disadvantage of installing a turbulence generating grid in the working section of the tunnel, where the flow velocity is highest and consequently the loss factor of the grid at its highest. However, this is the only practical way, given that the contraction ratio of 7.4:1 precludes installing them in the settling chamber, as the turbulence produced would be significantly damped by the contraction.

The balance used to record data in the model scale tunnel is a 6 component mechanical balance using strain gauges. The overall accuracy of the experiment, based on repeatability tests and the quoted accuracy of the balance, is better than 2 counts of drag ($\Delta C_D = 0.002$) between 15m/s and 47m/s (maximum tunnel speed).

FULL SCALE TUNNEL

Tests were conducted in the MIRA full scale wind tunnel using the MIRA reference car with a modified front end. The MIRA full scale tunnel is an open circuit, closed jet design with a working section size of 4.4m * 7.9m and a maximum speed of 36m/s. The balance in the MIRA tunnel is also a 6-component mechanical balance that uses strain gauges. The overall accuracy is quoted as better than 3 counts of drag ($\Delta C_D = 0.003$) at the

standard test speed of 28m/s. Wooden shapes were used to reduce the radius on the leading edge of the bonnet from 152mm to between 25mm and 100mm. The additional pieces extended across the front of the bonnet to the extent of the single radius so as to avoid the complication of the corners where three radii meet. Reynolds sweeps from 9 to 36m/s were used to identify the Reynolds sensitivity of the smaller radii, having previously established that the baseline MIRA reference model was not very sensitive to Reynolds number, in its estate car configuration.



Figure 2 Radius section installed on MIRA full scale reference model

The MIRA reference model is shown in *Figure 2* with one of the edge radius extension pieces added.

The Reynolds sweeps were carried out in the unmodified tunnel and with 2 different turbulence grids installed.

TURBULENCE GENERATION AND MEASUREMENT

In the model tunnel, turbulence was initially generated using standard square grids made of welded steel wire. The grid size was 50mm and the wire diameter was 2.41mm for the first grid and 3.85mm for the second, giving blockage ratios of 9.6% and 15.4%. The grids were mounted 1.5m upstream of the model front face, giving a distance of 30 mesh diameters between the grid and the model (as used by Wiedemann and others [8, 13]). The turbulence was measured using a CTA hotwire system. A vertical line traverse at the model centre gave average turbulence intensities of 1.38% for the first grid and 1.45% for the second. The clean tunnel turbulence intensity was measured as 0.18%.

Two further grids were devised, using masking tape on the second wire mesh to raise the blockage ratio. Several schemes were considered and tested using the hotwire to check for uniform turbulence intensity and average flow velocity across the test section. *Figure 3* shows the grid pattern chosen. The blockage ratio was

varied by using two different tape sizes to create the two turbulence grids. Using 25mm tape gave a blockage ratio of 37% and turbulence intensity of 5.13%. Using 12mm tape gave a blockage ratio of 24.5% and a turbulence intensity of 3.41%. The tape scheme was designed to be symmetric about the tunnel floor to avoid the problem of velocity overshoot in the boundary layer.

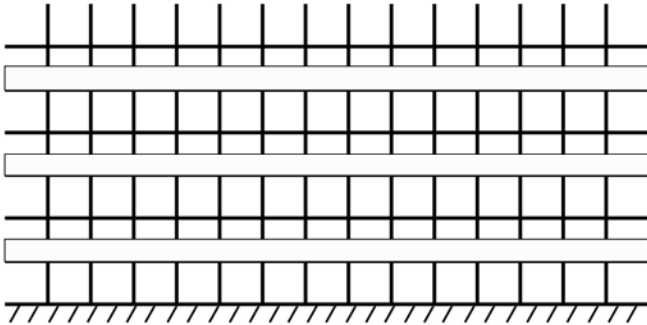


Figure 3 Bottom of grid showing 25mm masking tape scheme

The length scales were calculated by fitting the Von Karman spectral model to the measured data, as described by Li and Melbourne [14]. This gave small length scales for all the grids, as would be expected. The 1.38% grid had a length scale of 16mm, the 1.45% grid 15mm, the 3.47% grid 24mm and the 5.13% grid 37mm. The length scale for the clean tunnel was calculated using this method but is considered of little use, as the value relates to the length of the circuit rather than the size of any turbulent eddies.

In the full scale wind tunnel, turbulence generating grid patterns were installed at the entrance to the contraction on the upstream edge of the honeycomb section flow straightener. The honeycomb in the MIRA tunnel is large, with diamond sections approximately 300mm across and 1m deep, making it ideal for supporting the large grids, whilst unlikely to significantly reduce the turbulence generated. Several grid schemes were considered, and two schemes were chosen to use for the tests. Both consist of 9 wooden strips mounted vertically at a spacing of 1.4m across the honeycomb, with the spacing reducing to 0.7m between the pieces of wood at the edges and the walls to give symmetry as before.

The first grid uses 200mm strips, and the second 300mm. Figure 4 shows the second of the grids from behind the model position with the traverse on the main turntable. The first grid was found to generate a turbulence intensity of 3.4%, and the second 4.3%. Hotwire measurements of the standard tunnel showed it to have a turbulence intensity of 1.8%. As a consequence of the practicalities of installing such a large grid, only vertical bars were used, which may have the fortunate result of being more representative of the type of low-level turbulence measured by Richards [2]. Normally grid generated turbulence is expected to decay towards isotropic and homogeneous turbulence, but this

is generally claimed for grids with both horizontal and vertical bars. It may be that using vertical bars generates more longitudinal and lateral turbulence and less vertical, but it was not practical to test this. The homogeneity of the turbulence was measured by a traverse at the model location and found to vary by $\pm 0.3\%$ for both the grids and the clean tunnel. The length scales given using the Von Karman method were 160mm for the 3.4% grid and 165mm for the 4.3% grid. The length scale for the clean tunnel was 250mm. For the full scale tunnel, the grids are upstream of a 1.3:1 contraction, which would be expected to affect the length scales and intensity.

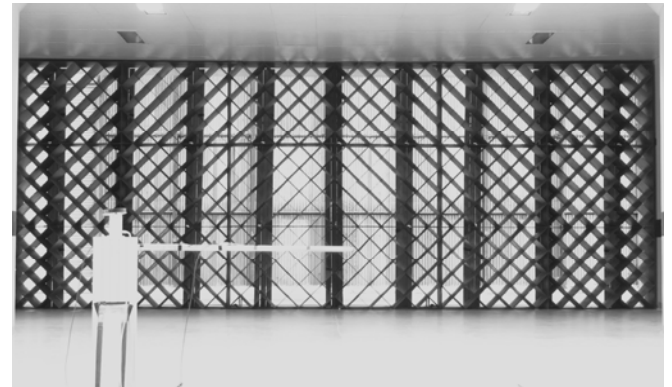


Figure 4 Full scale tunnel grid viewed from the model position

RESULTS AND DISCUSSION

MODEL SCALE RESULTS

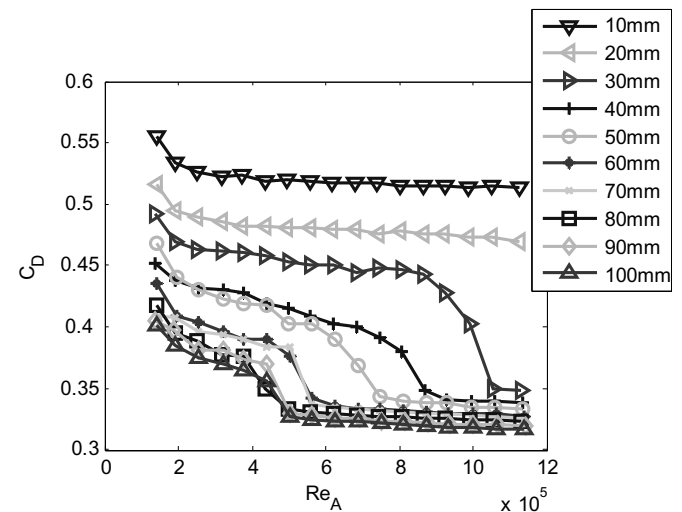


Figure 5 Variation of drag coefficient with Reynolds number based on the square root of frontal area in the clean tunnel (edge radius as a parameter)

Figure 5 shows the results of Reynolds sweeps in the clean tunnel for the drag coefficient (C_D). All but the smallest two front sections undergo a transition from high C_D at low Reynolds numbers to low C_D at high Reynolds numbers flow. This indicates that there is

probably some separation around the radius and over the bonnet for each front section, which disappears at higher Reynolds number. The separation is sustained as the Reynolds number increases and then the C_D drops over a small Reynolds number range to a consistent lower value. The fact that all the radii – apart from 10mm and 20mm, which never drop from the initial high drag state – show this characteristic would appear to indicate that there is some additional interaction between the leading edge radius and the windscreen, in comparison with the 1-box model, where only the smaller radii showed this shape. The larger radii on the 1-box model showed a high initial C_D which dropped in a smooth curve to an asymptotic consistent C_D at higher Reynolds number.

In previous papers the results were non-dimensionalised against the square root of frontal area, but this has not been done here, because it does not aid comparison between the full scale and model scale results. Instead the comparisons focus on the local Reynolds number based on edge radius.

Some of the characteristics of the curves are picked out in *Figure 6*, which shows the 40mm front section on its own for clarity. Based on Cooper's wool tuft testing [12] as well as data on bonnet and windscreen flows as reviewed by Hucho [15], it is believed that in the low Reynolds number flow, the laminar boundary layer separates from the leading radius, probably reattaching down the sides of the model and on the windscreen or the bonnet. As the Reynolds number increases the reattachment point moves along the sides towards the front, reducing the size of the separation bubble on the sides.

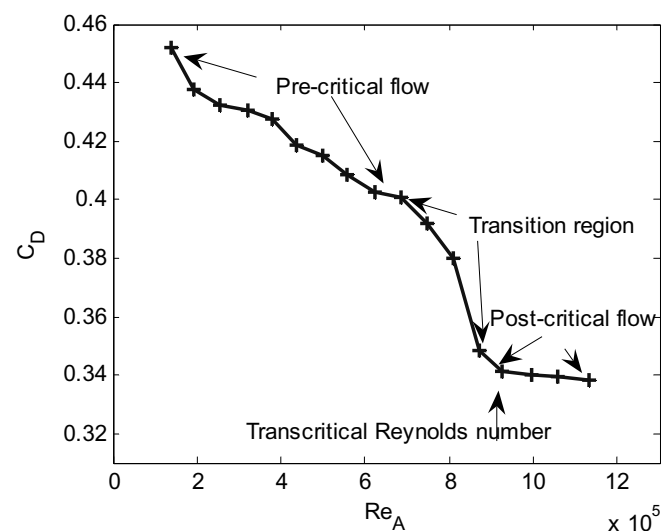


Figure 6 Variation of C_D with Reynolds number Re_A ($\eta=0.104$ only) showing curve characteristics

The reattachment point on the top surface also moves further towards the front of the car, resulting in the gentle reduction in drag coefficient. The separation bubble is believed to be largely stable at this point with a laminar separation and initially laminar free shear layer. At a

high enough Reynolds number the boundary layer becomes fully turbulent prior to separation, leading to fully attached flow over the body, possibly with a small separation bubble remaining for the smaller radii, where even a turbulent boundary layer does not stay attached around the radius. It is intended to back up these assumptions by making detailed flow measurements around the radii as part of the further work.

The transcritical Reynolds number as indicated on *Figure 6* is defined as the point where C_D falls to a low and approximately constant value after the critical Reynolds number. The critical Reynolds number is defined as the point where the separated laminar boundary layer becomes unstable and begins to undergo transition to turbulent flow (Hoerner [16]). While it is possible to identify the transcritical Reynolds number by eye for many of the radii, for consistency and ease of analysis an algorithmic method was defined. A linear best fit of the last points recorded was identified, and then, using a "Piecewise Cubic Hermite Interpolating Polynomial" curve fit, the point where the curve deviates significantly from the line was picked out numerically. The PCHIP fit was used because it produces a result that closely fits the points with little overshoot.

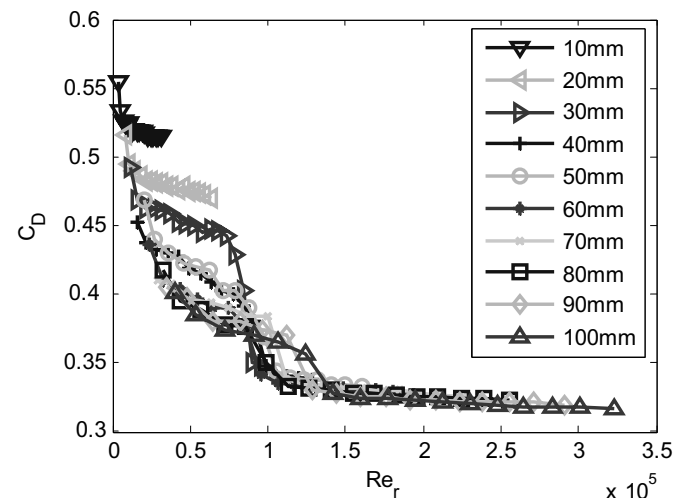


Figure 7 Variation of drag coefficient with Reynolds number based on edge radius in the clean tunnel (edge radius as a parameter)

Figure 7 shows the results with Reynolds number based on edge radius rather than the square root of frontal area. This shows that the transcritical Reynolds numbers based on edge radius are reasonably constant, in agreement with Cooper [12] and Newnham [11]. The figure also shows that the maximum tunnel speed does not result in a high enough local Reynolds number for the 10mm and 20mm front sections to undergo transition.

The average transcritical Reynolds number based on edge radius, $Re_{r,crit}$ determined from the data in *Figure 7* is 1.07×10^5 . This compares well to the previous

published values, for 1-box models, of 1.30×10^5 by Cooper [12] and 1.24×10^5 by Newnham [11].

clean tunnel results are not shown in the turbulent case, and the progression from high C_D at low Re_A to low C_D at high Re_A is much smoother.

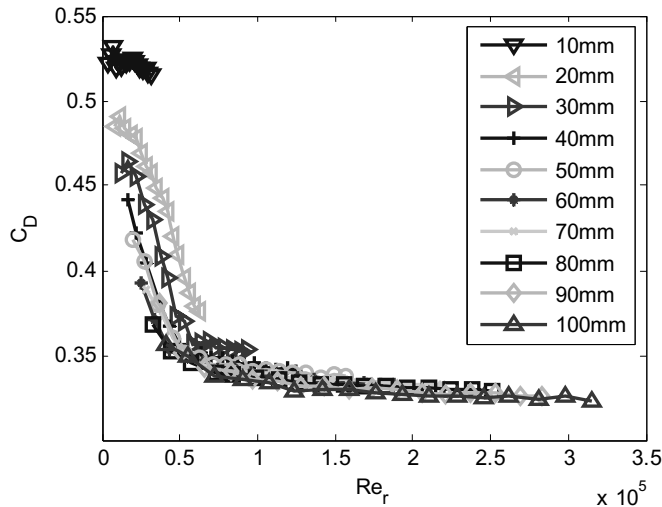


Figure 8 Variation in drag coefficient with Reynolds number based on edge radius for the first turbulence grid ($Tu=1.38\%$) (edge radius as a parameter)

Newnham [11] found for the 1-box model that when the turbulence intensity was raised to 1.4% the transcritical Reynolds number was reduced. Figure 8 shows that this is also true for the 2-box model. The transcritical Reynolds number drops to an average of 0.56×10^5 .

As the constant Re_{crit} has been reduced, the 20mm front section now undergoes transition, although it does not exhibit a full transition to a consistent low C_D value. The 10mm front still does not undergo transition, because the maximum local Reynolds number is lower than the transcritical value.

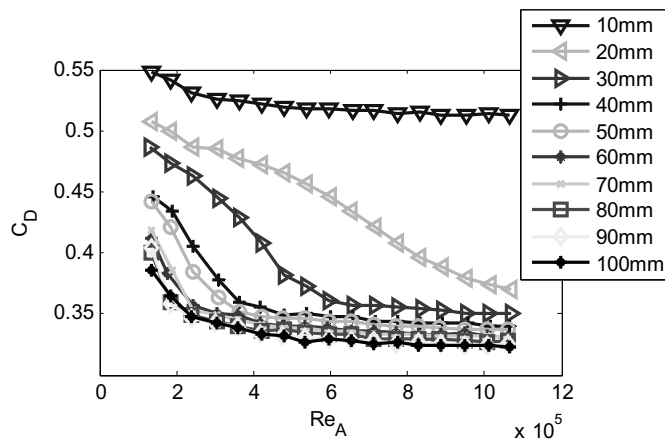


Figure 9 Variation in drag coefficient with Reynolds number based on frontal area for the second turbulence grid ($Tu=1.45\%$) (edge radius as a parameter)

Figure 9 shows the results for the 1.45% turbulence grid plotted against Reynolds number based on frontal area. Comparing this figure with Figure 5 shows how the characteristics of the curves have been changed by the additional turbulence. The sharp drops shown for the

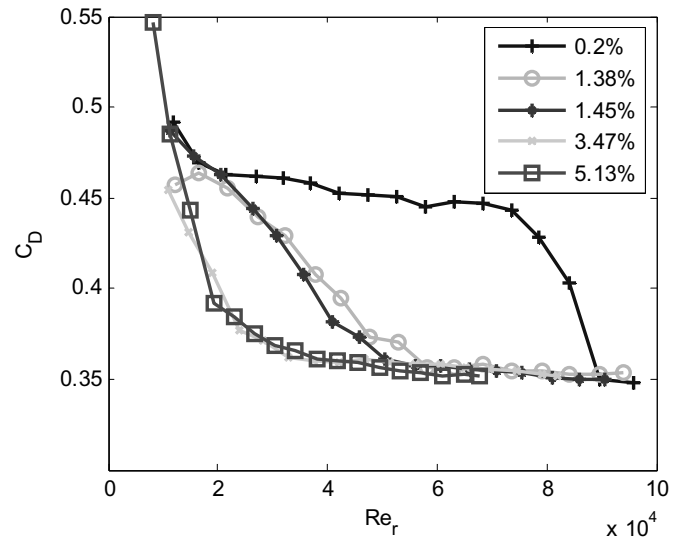


Figure 10 Variation of drag coefficient for the 30mm front section only (turbulence intensity as a parameter)

Figure 10 shows the results for the 30mm front for all the turbulence conditions tested. This is the smallest radius that underwent transition in the clean tunnel, and it can be seen that the transcritical Reynolds number reduces with increasing turbulence up until the 3.47% level, after which there is little further reduction. The shape of the curve also changes under the different turbulence conditions. In the clean flow, the transition is very abrupt. With around 1.4% turbulence, transition is gradual, taking place over around half the Reynolds number range. With higher turbulence, transition occurs very quickly and the curve then drops very slowly towards the minimum value.

The additional freestream turbulence appears to have the effect of increasing the boundary layer growth, inducing the boundary layer to undergo transition to turbulent at a lower Reynolds number. This is backed up by Bearman [6] who in addition states that the freestream turbulence also increases the likelihood of a free shear layer undergoing transition to turbulence.

At low Reynolds numbers the addition of turbulence is seen to reduce the drag coefficient but above the transcritical value, the drag coefficient is higher by a relatively smaller amount. This matches expectations based on the results of Cogotti and Wiedemann [8, 9].

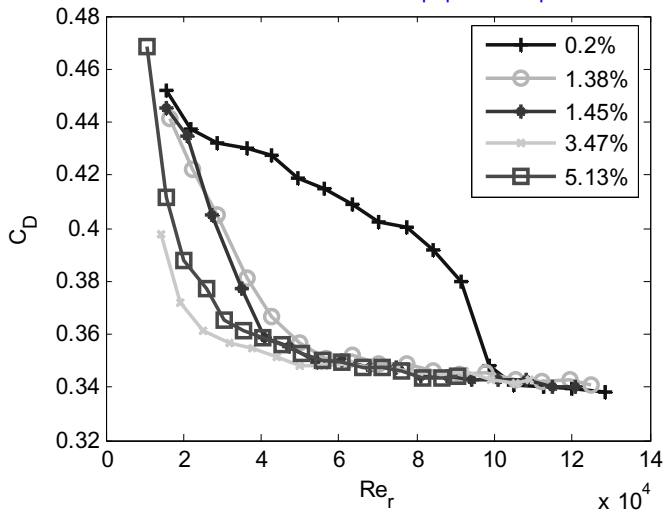


Figure 11 Variation of drag coefficient for the 40mm front section only (turbulence intensity as a parameter)

The results for the 40mm front only are shown in Figure 11. The results are similar to Figure 10, but the larger radius means that all the transcritical Reynolds numbers based on edge radii occur at lower speed, shifting them towards the left hand edge of the figure (although they remain numerically constant). This trend is repeated for all the radii. The trend with turbulence intensity is less clear. For the 30mm front section in Figure 10 the turbulence grid results are in rough order from lowest to highest, but in Figure 11 the 3.47% grid gives the lowest result for drag coefficient at low Reynolds number. The reason for this is not well understood, but it is expected that the turbulence intensity affects different parts of the flow over the model differently. For example, Bearman [6] and others have shown an effect of turbulence on base drag, which has not been investigated here.

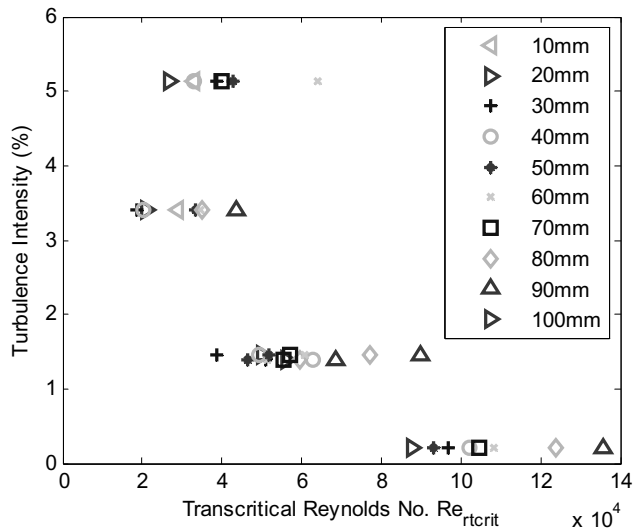


Figure 12 Variation in transcritical Reynolds number with turbulence intensity (η as a parameter)

The transcritical Reynolds numbers for all the front sections that show a clear transition point are shown in Figure 12, replotted to show how $Re_{r,crit}$ varies with

turbulence intensity. The figure shows that there is a large difference between the results for 0.2% turbulence and 1.4% turbulence, and then a smaller difference to 3.4% turbulence, and almost no difference to 5.1%.

The graph indicates that the average transcritical Reynolds number begins to rise between 3.4% and 5.1% turbulence, suggesting that, for the range of intensities tested, there may be a minimum value of $Re_{r,crit}$ at model scale.

The post-critical drag coefficients, at the highest Reynolds number tested, show that the drag with turbulence is generally higher than without. Figure 13 shows the variation in post-critical drag at the highest Reynolds number tested for the 5.1% turbulence grid. Only the 60mm front radius and above are shown, because they are the only ones where measurements of the post-critical condition were taken at the same Reynolds number based on the square root of frontal area.

The post-critical C_D is consistently lower in the clean tunnel for any given radius, and almost always highest in the highest turbulence case. However, the 1.38% turbulence grid produces a very large rise, and then between 1.38% and 1.45% there is a large drop. This large difference is not well understood, because the length scales (15mm and 16mm) are also quite similar. There could be some effect on the base drag, as this has been noted by Bearman[6] and others. In general, turbulence does seem to increase the drag coefficient, as expected from Cogotti's results[9], but these results do not provide a predictable trend.

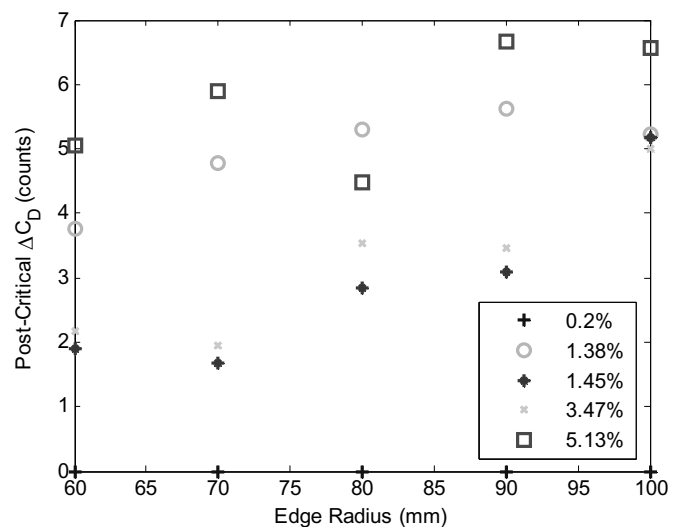


Figure 13 Change in post-critical C_D relative to $Tu=0.2\%$ with edge radius at $Re_A=8 \times 10^5$ (Turbulence case as a parameter)

FULL SCALE RESULTS

Figure 14 shows the C_D results for all radii against Reynolds number based on the square root of frontal

area, as plotting in this form allows the characteristic shapes to be more clearly identified. As expected, the smaller radii results are dependent on Reynolds number, with less variation in C_D than in the model scale tunnel. The MIRA reference model is different and also the radius was only changed for one of the leading edges. These results are similar to the results for the 1.45% turbulence results (Figure 9) at model scale, in terms of the transcritical Reynolds numbers and general shape of the curves. Since the turbulence intensity is slightly higher at full scale, the transcritical Reynolds numbers are lower, and the pre-critical flow less evident.

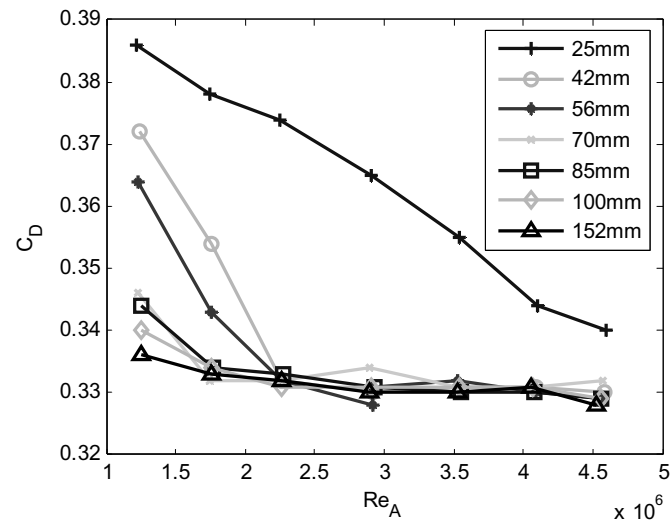


Figure 14 Variation of drag coefficient with Reynolds number for the unmodified full scale tunnel (edge radius as a parameter)

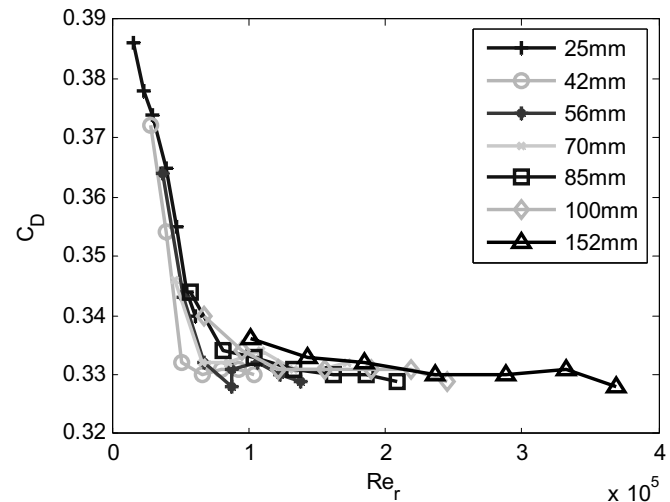


Figure 15 Variation of drag coefficient with Reynolds number based on edge radius in the unmodified full scale tunnel (edge radius as a parameter)

As in the model scale, Figure 15 shows the results replotted against Re_r , and it is clear that they are similar in shape to Figure 8, but with much less variation in C_D . The range of Reynolds number tested is larger overall, but the smallest radius does not show fully post-critical

flow, and the largest does not enter the transition region at the low end. This was because the wind speed range is more restricted at full scale, to between 9.5m/s and 36.0m/s. There is still a clear demarcation between the transition region and the post-critical flow.

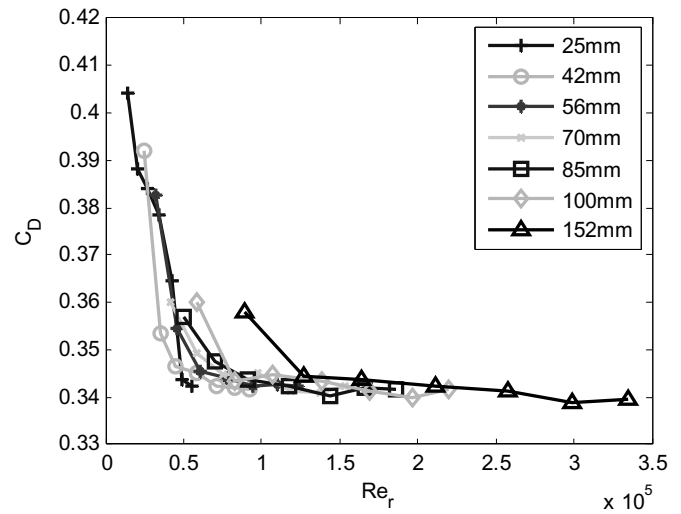


Figure 16 Variation of drag coefficient with Reynolds number based on edge radius for the 3.4% turbulence grid (edge radius as a parameter)

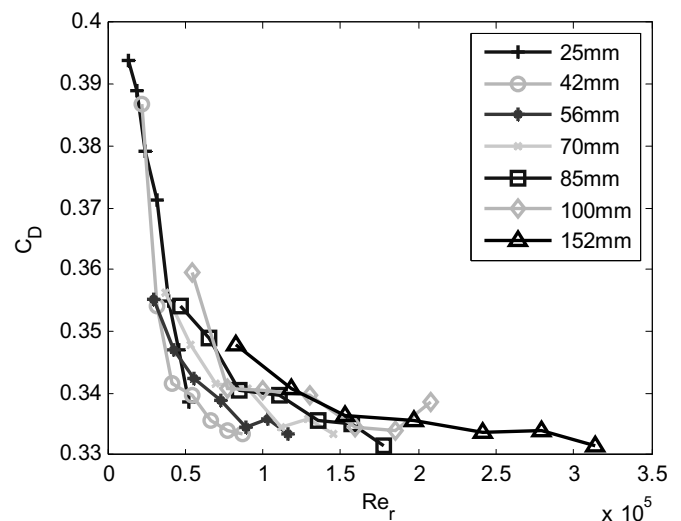


Figure 17 Variation of drag coefficient with Reynolds number based on edge radius for the 4.3% turbulence grid (edge radius as a parameter)

With the turbulence grids installed, the turbulence intensity was similar to that for the 3rd and 4th grids in the model scale tunnel. Figure 16 and Figure 17 show the results for the turbulence grids against Reynolds number Re_r . As could be expected from the model scale results, there is some reduction in the transcritical Reynolds number, but not a dramatic difference compared to Figure 15. Ignoring the two largest radii, which do not show a convincing transition region (only the lowest point, where the tunnel speed is at its least stable, is at all higher than the others), Figure 16 shows a very tight

group of transcritical Reynolds numbers for the various radii, while the spread is larger for *Figure 17*.

which is known to be affected by turbulence (eg. Bearman [6]).

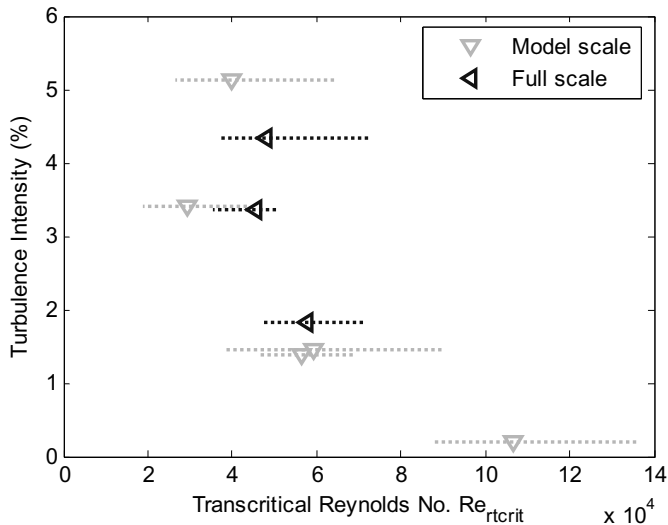


Figure 18 Variation in average transcritical Reynolds number with turbulence intensity for both wind tunnels

The transcritical Reynolds numbers were extracted from the full scale data using the same method as described earlier. *Figure 18* shows the average results at each turbulence level plotted against turbulence intensity. The dotted lines indicate the full spread of the results. It is immediately clear that there is good agreement between the model scale and full scale results. The transcritical Reynolds number again reaches a minimum and then rises slightly at the highest turbulence intensity.

It is worth noting that only 5 of the 7 radii tested at full scale showed convincing transcritical Reynolds number, and it is suggested that a useful extension to this work would be to increase the number of radii tested in the range that do go through the transcritical Reynolds number within the speed range tested.

Figure 19 shows a comparison of the results for the 42mm front section only. There is some variation in the shape of the curve between the unmodified tunnel and the turbulence grid results, in a similar manner to the difference between the results for the ~1.4% turbulence grids and the higher turbulence grids at model scale (*Figure 11*).

In order to ensure correct comparison between the C_D values, the reference pitot was calibrated against the tunnel centreline with and without the turbulence grids installed, and the results were corrected based on this calibration. The results show that raising the turbulence level above the clean tunnel level did increase the drag, but going between 3.4% and 4.3% actually reduced it again. The difference between the baseline and the 3.4% grid result is an increase of 10 counts of drag. The 4.3% grid then reduces the drag by as much as 8 counts from that level, bringing it very close to the clean tunnel result. The reasons for this are not well understood, but could be as a result of changes in the base pressure,

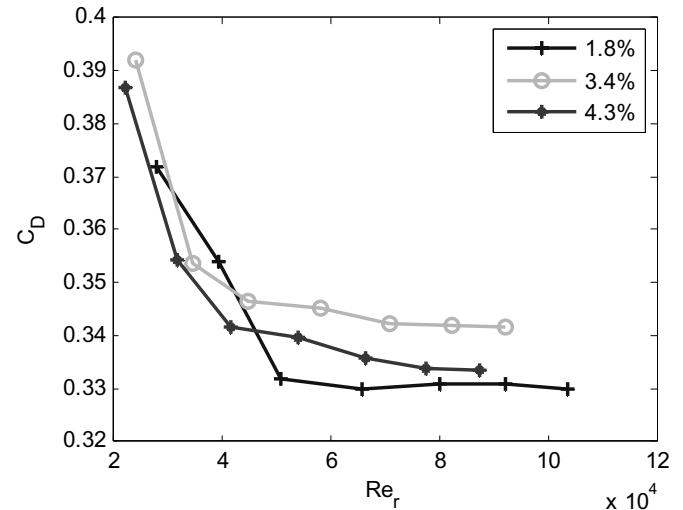


Figure 19 Variation of drag coefficient with Reynolds number, 42mm front radius only (turbulence intensity as a parameter)

FURTHER WORK

It is intended to extend this work to cover the effect of turbulence on rear shaping, making balance measurements on a model with an adjustable backlight angle and comparing the results at different levels of turbulence, and with previous studies.

It is also intended to make further measurements on the current models, looking at the flowfield around the radii with a hotwire and with an LDA system. The state of the boundary layer and its thickness will be examined for several radii at significant Reynolds numbers.

CONCLUSION

The effect of turbulence on the aerodynamic drag of a simple 2-box shape with different front edge radii has been investigated for a basic model and a generic car shape at full scale. Both model and full scale tests show a significant reduction in the transcritical Reynolds number for a given edge radius with increasing turbulence.

The model tests show that overall drag coefficient is noticeably increased with the introduction of a small level of turbulence and increases slightly with higher turbulence. The raised turbulence level also caused the shape of the Reynolds sweep curve to change. In the very low turbulence of the clean tunnel, the drag coefficient persisted at a high value up to a higher Reynolds number, followed by an abrupt collapse. When the turbulence intensity was increased, the transition to low drag post-critical flow was much more gradual.

The full scale results showed a good correlation with the model scale results for the variation of transcritical

Reynolds number based on edge radius with turbulence intensity. The drag coefficient was also higher with the grids installed, although the high turbulence grid produced a lower C_D than the lower turbulence grid.

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DEFINITIONS, ACRONYMS, ABBREVIATIONS

η	non-dimensional edge radius
A	frontal area
C_D	drag coefficient
r	front edge radius
Re_A	Reynolds number based on the square root of frontal area
Re_r	Reynolds number based on front edge radius
Re_{rcrit}	transcritical Reynolds number based on front edge radius
Tu	turbulence intensity