

## 1 Article type

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## 8 Article title

9 On the dynamic response of an instrumented headform for alternative mounting  
10 stiffnesses when subjected to ballistic impacts

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## 19 Abstract

### 20 Abstract

21 The current British Standard for head protectors for cricketers has been recently revised to include a  
22 projectile-based battery of tests, the intention being to ensure that a certified helmet will also prevent contact  
23 of the ball or grille with the specified headform facial region. The purpose of this study was to characterise  
24 the dynamic response of the headform to direct ballistic impacts for alternative headform mounting  
25 arrangements. On the one hand, and in accordance with the relevant sections of the Standard, what might  
26 be described as a “Constrained” set-up was evaluated while, on the other, an arrangement with significantly  
27 reduced stiffness, in line with that previously reported for the passive human neck, was subject to equivalent  
28 appraisal.

29 For each mounting scenario, an air cannon was used to project a cricket training ball at three speeds toward  
30 the instrumented headform at three locations with five repeats per speed/location combination. High-  
31 rate/resolution video and piezo-electric accelerometer data were collected and processed to determine the  
32 headform response. While differences between specific ball impact speed and location scenarios are set out  
33 in detail later in the article, overall observations are summarised as follows. From a ball-headform contact  
34 duration standpoint, video derived results showed ranges of 1.30 – 1.45 ms (Constrained) vs. 1.26 – 1.41  
35 ms. Maximum ball deformations the timing of which enabling the event to be subdivided into “loading” and  
36 “unloading” phases, were found to be 82.5 – 86.2% (Constrained) vs. 82.8 – 86.4% of original ball diameter,  
37 mean peak headform accelerations during loading were found to be 860 – 1615 m/s<sup>2</sup> (Constrained) vs. 967  
38 – 1638 m/s<sup>2</sup> while headform speeds at the end of the loading phase were found to be 0.5 – 0.92 m/s  
39 (Constrained) vs. 0.54 – 0.93 m/s. Differences between headform response for the two mounting  
40 arrangements were observed to be more substantial during the loading rather than unloading phase.

### 41 Keywords

42 Sports safety, British Standards, instrumented headform, ballistic impacts, dynamic performance

43

1 On the dynamic response of an instrumented headform  
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30 rather than unloading phase.

## 1 Introduction

2 Head protection is commonly available in many sports. A participant's use of a helmet  
3 may be mandated in the rules of a competition, for example in Formula 1<sup>TM</sup> motor racing<sup>1</sup>  
4 and snowboarding<sup>2</sup>. In some sports, however, helmet use may be permitted in the rules,  
5 but ultimately will be a personal decision aimed at reducing the risk of incurring injury.  
6 Until recently this latter scenario was the case in cricket<sup>3</sup>. However, largely in response to  
7 recent changes to the nature of the game<sup>4</sup> and most certainly one particularly tragic  
8 event<sup>5</sup>, the England and Wales Cricket Board has made the use of a helmet mandatory for  
9 batsmen from the 2016 season and onwards<sup>6</sup>. In extreme cases, the function of a helmet is  
10 to prevent death or life-changing injury, although, in many cases, the role of a helmet is  
11 to protect against more minor injury, disfigurement or discomfort.

12 Walker et al.<sup>7</sup> reported that, over a five-year period in New Zealand, 21% of all injuries  
13 resulting in hospitalisation for professional and recreational cricketers were sustained to  
14 the head. In elite-level cricket, the speed of the ball can exceed 90 mph when bowled or  
15 thrown<sup>4</sup>, or, of course, higher still when hit by the bat. While lower speeds are inevitably  
16 more normal at recreational levels of the game, direct contact between the ball and the  
17 head or contact as a result of helmet deformation can still cause injuries, such as  
18 concussions, eye injuries, facial fractures, lacerations<sup>8</sup> or even death<sup>9,10</sup>. Safety is not, of  
19 course, the only consideration when developing (sports) helmets; comfort and the linked  
20 ability of the wearer to effectively perform are also highly important factors<sup>11</sup> with  
21 designers and engineers typically having to compromise. In cricket specifically, a strong  
22 tradition further results in the styling and aesthetics also being of importance. Indeed, a  
23 helmet providing complete, all-round protection for a cricket batsman is unlikely to be

1 utilised should it restrict vision, prohibit optimal performance or drastically differ from  
2 the traditional appearance of cricket headwear.

3 To be sold legally (in Europe), protective helmets must satisfy current standards, and the  
4 tests therein, and thereby adhere to certain quality controls<sup>12</sup>. In the specific context of  
5 cricket helmets (for batsmen), the scope for the relevant Standard, BS 7928: 2013<sup>13</sup>,  
6 includes specification of “*the requirements for the materials, construction, markings and*  
7 *information to be supplied for head protectors*”. Additionally, and with appropriate  
8 reference to other generic protective helmets performance test standards<sup>14</sup>, BS 7928: 2013  
9 “*specifies the methods to assess the impact attenuation properties during a drop test of*  
10 *the helmet and the protection provided against a ball or faceguard contacting a specified*  
11 *no contact zone of the face during a projectile test*”. Such tests must represent an  
12 appropriate performance benchmark at reasonable cost to avoid excessive development  
13 costs being passed on to the consumer. For these reasons, relatively simple mechanical  
14 arrangements, often based on energy equivalent collisions between objects of increased  
15 mass at reduced velocities (such as the free-fall test described by Johnson<sup>15</sup>) are typically  
16 favoured. This is largely due to the relative ease with which i) controlled, repeatable tests  
17 can be conceived and executed, ii) the necessary observations can be made and iii) the  
18 pass/fail criteria can be defined; product performance can thereby be confidently  
19 evaluated.

20 The accepted impact attenuation performance battery of tests within BS 7928: 2013  
21 follows such a path. Here a falling (magnesium alloy<sup>16</sup>) headform method<sup>14</sup> is proposed  
22 in which “*A steel anvil, with a hemispherical striking face with a diameter of (73 ±1)*  
23 *mm*”<sup>13</sup> is specified. The required impact speed (for a size 575 headform) must be 2.53-

1 2.63 m/s and the resultant deceleration, “*measured by a tri-axial accelerometer located*  
2 *within the headform*” of the helmeted headform, must be no more than 250 g. Four  
3 impact sites per helmet type must be tested with a single impact conducted per helmet  
4 sample at each impact site.

5 Such simplified equivalents have, however, been shown to be less valid for the case of a  
6 ball projected at the gap between the peak (the protrusion just above the eye-line of the  
7 shell of the helmet) and the grille (face protector commonly made of metal) of a typical  
8 cricket helmet<sup>17</sup>. For this reason, it was a projectile, rather than a falling mass or  
9 headform test, that was ultimately incorporated into BS 7928: 2013; revised in response  
10 to recent high-profile injuries<sup>4</sup>. While maintaining the impact attenuation test element, the  
11 intention of the inclusion of the facial contact projectile test is to ensure that helmets are  
12 also capable of preventing facial contact, either directly from ball penetration of the peak-  
13 grille gap or as a result of deformation of the faceguard following ball impact. To enable  
14 this facial-contact projectile test, a “*ball, that has experienced no more than 20 impacts...  
15 [with] a diameter of between 71 mm and 73 mm... [and] mass... between 140 g and 150  
16 g for adult helmets... [is projected] at velocities up to (28 ±3) m/s... [to impact] no  
17 greater than 10 mm from the expected target location*”<sup>13</sup>. Five target locations are  
18 specified, three along the helmet centreline with two laterally. Determination of whether  
19 or not the ball or the grille has come into contact with the headform is determined by the  
20 use of contact indicator, e.g. “Developer” spray commonly used in crack/flaw detection.

21 Like the impact attenuation drop test, the facial contact projectile test also requires that  
22 the helmet be fitted to a headform albeit, in this case, without the inclusion of the  
23 accelerometer. Clearly the headform must be mounted by some means and there may be

1 debate over the exact manner in which to do this so as to optimise “biofidelity”, including  
2 whether or not a representation of the neck should be incorporated; the means by which  
3 this fitting should be achieved is not explicit in BS 7928: 2013. The purpose of this study  
4 was, therefore, to evaluate the behaviour of two near-extreme examples of headform  
5 mounting arrangement in terms of their influence on headform dynamics during and after  
6 an un-helmeted ballistic impact.

7 The relationship between headform dynamics (and the measures by which they can be  
8 defined for such an impact scenario) and the performance of a helmeted headform in  
9 terms of preventing facial contact may be questioned. The intention of this study was,  
10 however, more fundamental in terms of aiming to establish whether differences exist  
11 between these two near-extreme arrangements during and shortly after ballistic impact  
12 for the sake of better informing future research and development as well as standard tests  
13 including both facial contact and impact attenuation projectile tests. The research  
14 described herein uses state-of-the-art processes that are arguably not at the disposal of nor  
15 practically applicable for typical test houses that determine whether or not cricket helmets  
16 are safe. This intention is not, however, to have future Standards adjusted to require the  
17 use of such processes but more to complement existing capabilities and glean additional  
18 insights in support of the good work already being done. This paper will, for the first  
19 time, report differences between ball/headform impact dynamics that result from  
20 alternative headform mounting and ball impact speed/location during sports relevant  
21 impacts, specifically those typical to cricket.

22

# 1 Methodology

## 2 Experimental arrangement and data collection

3

4 The bespoke experimental arrangement employed in this study allowed for the  
5 suspension mounting of a BS EN 960: 2006<sup>16</sup> instrumented magnesium alloy headform,  
6 size 575 (mass 4.7 kg) in two alternative scenarios, as shown in Figure 1. In the,  
7 “Constrained” (deliberately not rigid because some compliance is inevitable and, indeed,  
8 desirable to limit the high shock when the impact occurs) scenario shown in Figure 1a,  
9 the headform was post-mounted via a clamp arrangement incorporating a stiff anti-  
10 vibration type bushing onto a grounded extruded aluminium frame. The bushing, shown  
11 in detail in Figure 1c, includes a silicon rubber element which enables constraint of the  
12 headform, while allowing some rotation about a pivot, the axis of which is 105 mm from  
13 the base of the headform.

14 In the alternative, “Unconstrained” scenario, the headform was inverted and suspended  
15 with bungee cords (10-mm diameter, 35-N pre-test tension) attached to the base as per  
16 Figure 1b. Attachment and alignment of the bungees require compromise with the ideal  
17 scenario being one in which the suspension is aligned normal to the direction of travel  
18 such that, upon disturbance from rest, there is no immediate bungee extension and,  
19 therefore, no immediate restoring force. This compromise comes, however, at the  
20 expense of readily repeatable positioning of the headform prior to each impact event and  
21 some deviation from this ideal is therefore required<sup>18</sup>.

22 The absolute stiffness values of the two suspension scenarios were experimentally  
23 determined by applying a series of torques to the headform while measuring the resulting  
24 angular displacement. Practically this determination of suspension stiffness was achieved

1 by applying a measured force, initially along the z-axis, at the reference plane (see Figure  
2 2a) while measuring the resulting angular displacement ( $\alpha$  in Figure 2a) using image  
3 processing. As angular displacement occurred, the line of action of the applied force was  
4 adjusted so that it maintained a direction that was in-line with and directly along the  
5 reference plane. For the Constrained scenario the stiffness was determined to be 5.6 N-  
6 m/deg, while, for the Unconstrained, a corresponding value of 0.2 N-m/deg was  
7 established. This latter value is within the range of the passive stiffness of the human  
8 neck, 0.03 – 0.3 N-m/deg<sup>19</sup>.

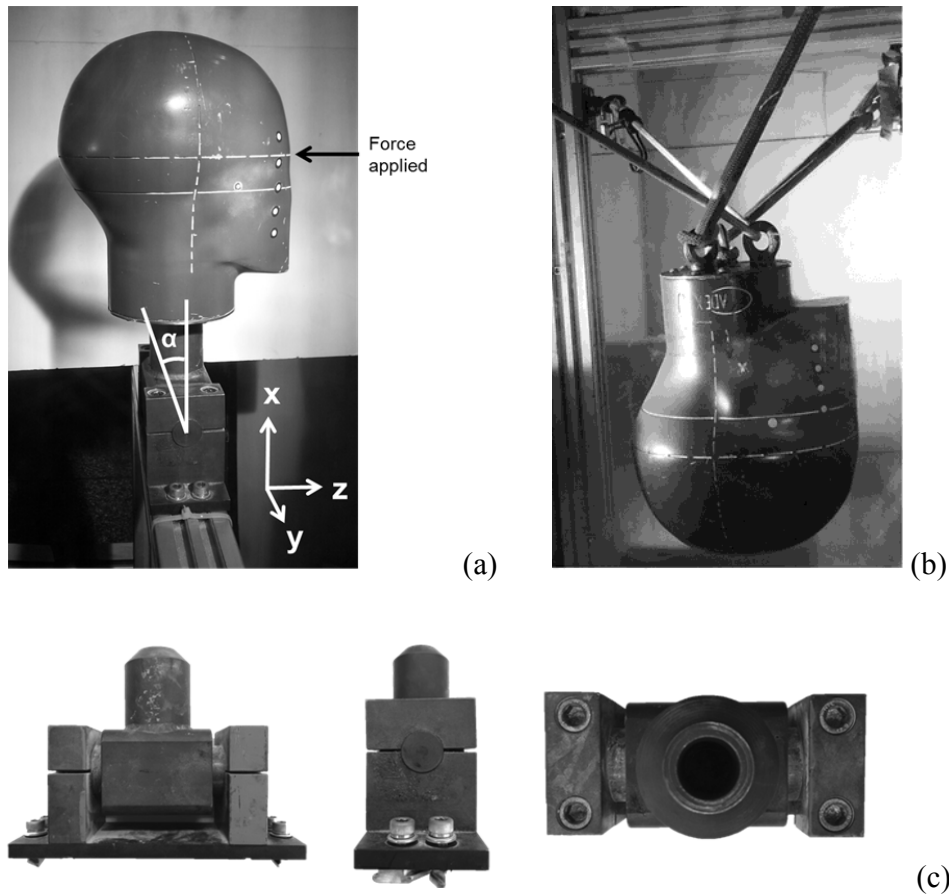
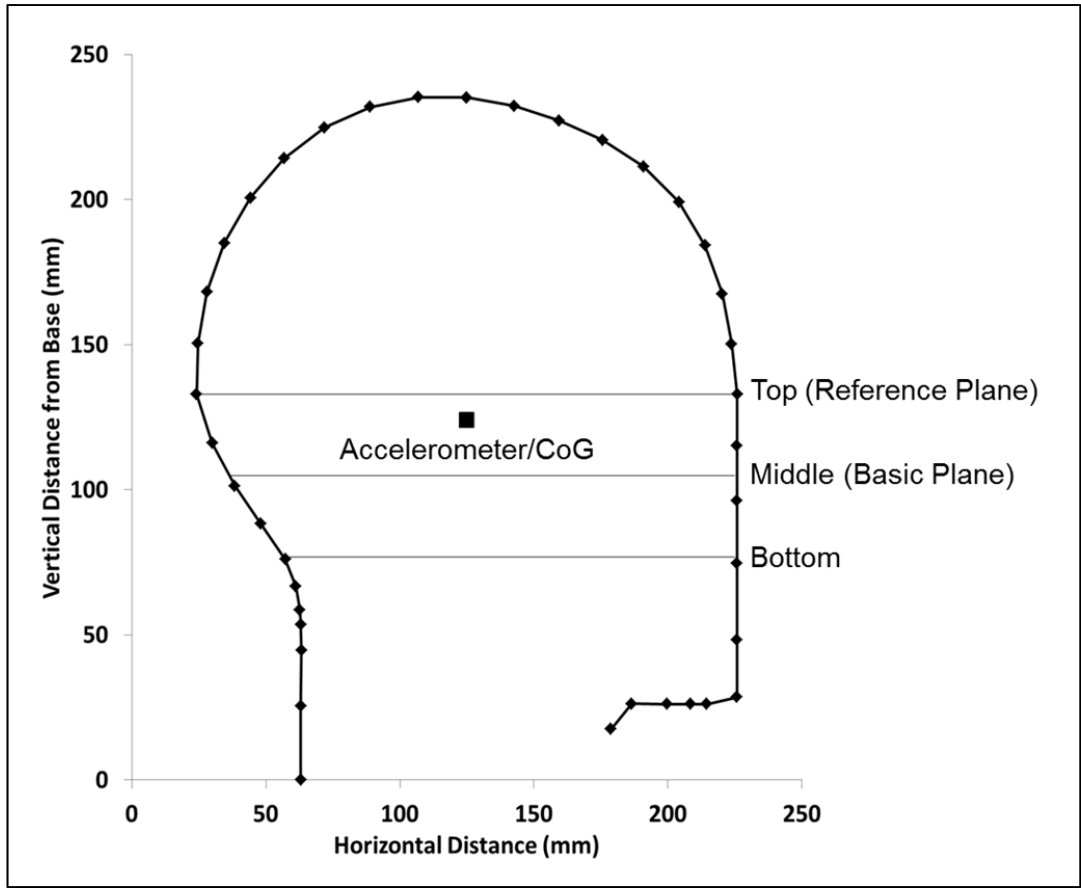


Figure 1. Alternative headform mounting arrangements; (a) “Constrained” via bushed mounting to a grounded frame, (b) “Unconstrained” with bungee cords between base and frame and (c) detailed images of the Constrained arrangement mounting.





1

2 Figure 2. Section view of headform through the sagittal plane with “Top”, “Middle” and  
 3 “Bottom” impact locations indicated as well as the accelerometer location.

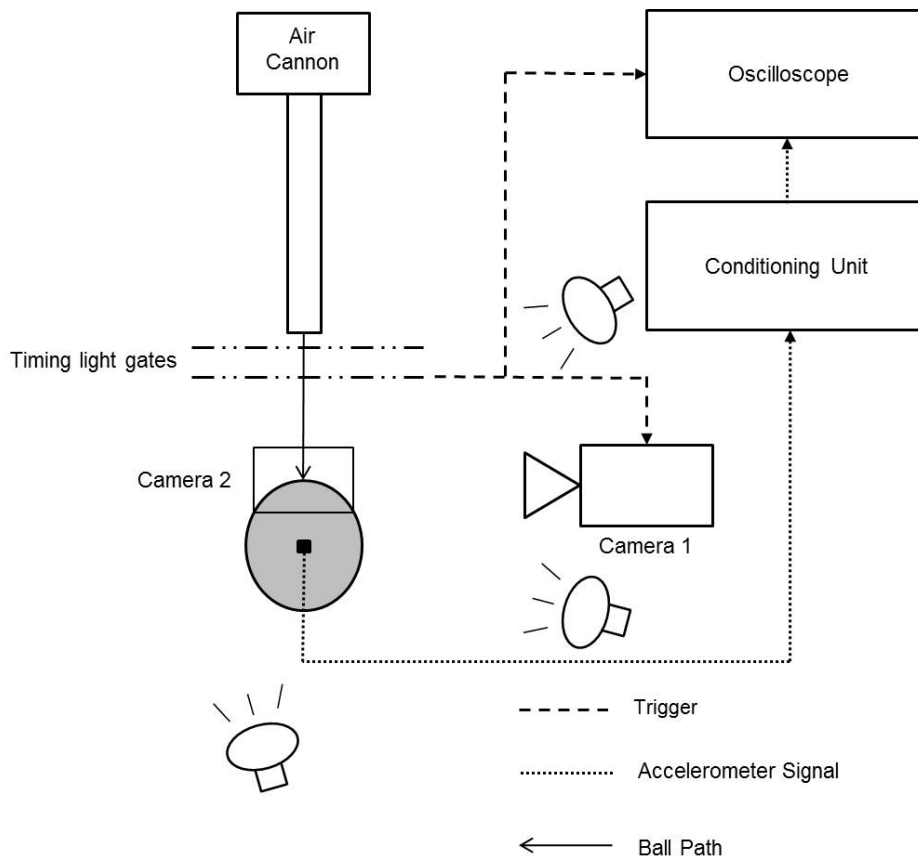
4 Throughout all tests the same type of “BOLA”™ cricket training ball was used (mass  
 5 nominally 150 g, diameter nominally 71 mm), as specified in BS 7928: 2013<sup>13</sup> for the  
 6 Facial contact projectile test. A pressurised air cannon was used to project the ball toward  
 7 the headform impact location at three speeds, nominally 28, 25 and 22 m/s, the first two  
 8 also in line with those specified in the Standard. The experimental arrangement allowed  
 9 for the vertical repositioning of the headform relative to the cannon such that impacts  
 10 could be arranged to occur at three locations, nominally 136 mm (“Top”), 108 mm  
 11 (“Middle”) and 80 mm (“Bottom”) from the base of the headform as shown in Figure 2.  
 12 The Top and Middle impact locations are aligned with the headform “Reference” and

1 “Basic” planes while the Bottom is the same distance from the Middle as the Top is but  
2 in the opposite direction. While the Top and the Middle impact locations and 28- and 25-  
3 m/s impact speeds are in line with the specified Standard, the Bottom and 22-m/s impact  
4 condition were included with a view to increasing any possibility to interpolate or indeed  
5 extrapolate from the results determined to other speeds and/or impact locations in the  
6 future.

7 It is worth noting at this point that, in the Standard, the helmeted headform positioning is  
8 specified such that the ball ideally makes contact with the peak and grille simultaneously,  
9 thereby maximising the chance of penetration occurring. Taking into account the variable  
10 design of commercially available helmets, this “simultaneous impact” scenario  
11 practically amounts to an impact location somewhere between the Reference and Basic  
12 planes, hence the choice of these two target impact locations. It is also worth noting that a  
13 helmeted headform, being more deformable upon ball impact, will result in an extended  
14 impact duration compared to an un-helmeted headform. This extended duration impact  
15 will include associated alternative characteristics including impulse and energy transfer  
16 although these may or may not lead to significant differences in equivalent performance  
17 to the scenarios investigated in this study.

18 A PCB 356B21 tri-axial piezoelectric accelerometer was rigidly fitted inside the  
19 headform using the headform manufacturer supplied stud mounting arrangement. The  
20 accelerometer was located at the headform centre of gravity (i.e. on the central axis,  
21 127 mm from the base) and orientated relative to the headform such that the three  
22 acceleration measurement components were aligned with the directions shown in  
23 Figure 1a. After the signal was conditioned using a DJB Instruments CV9 IEPE

1 amplifier, the accelerometer voltage signals were captured using a LeCroy WaveJet 324  
2 digital oscilloscope with a sample frequency of 50 MHz specified as such so as to yield  
3 substantial temporal resolution (with subsequent down-sampling if necessary readily  
4 possible) for the anticipated circa 1-2 ms duration impact events. Accelerometer  
5 sensitivities were determined in advance using a B&K Type 4294 accelerometer  
6 calibrator with the  $x$ -,  $y$ - and  $z$ -directions being 1.18, 1.22 and 1.17 mV/ms<sup>-2</sup>, respectively.  
7 Three Arri Pocket Par 400 lights were used to illuminate the testing area while a Photron  
8 FastCam SA1 mono high-frame-rate camera, operating at 50 kHz (448 x 224 pixels  
9 spatial resolution), was positioned lateral and perpendicular to the plane of movement, as  
10 can be seen in Figure 3 (Camera 1), 635 mm from the headform. This positioning  
11 allowed for a view of a portion of the headform and approximately 140 mm of the ball  
12 trajectory prior to and post impact as per the example extracted still images shown in  
13 Figure 4.

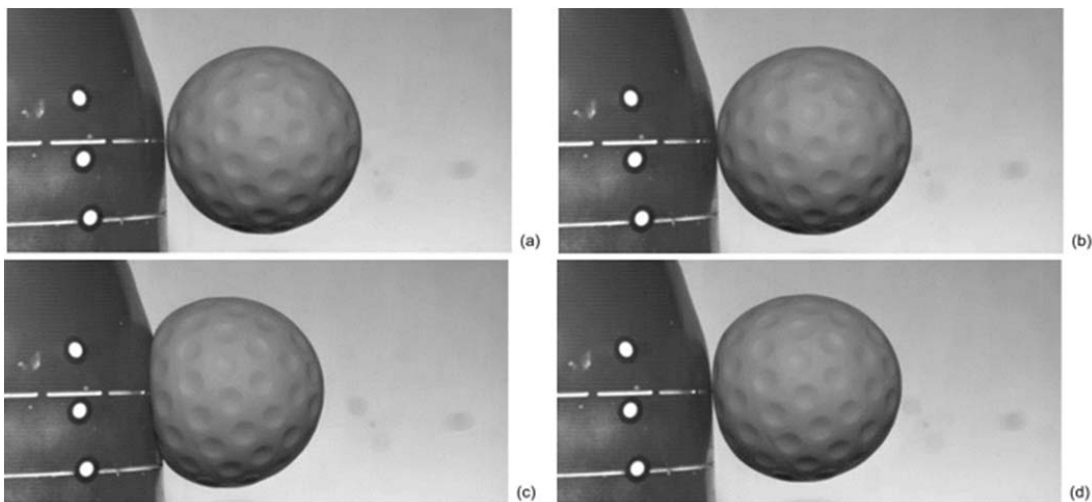


1

2 **Figure 3. Schematic of the experimental arrangement.**

3 Prior to each trial, the headform rest position was adjusted, making use of the video  
 4 camera images to compare the current with previous position, as required to achieve the  
 5 desired ball-impact location. While this adjustment was always necessary for the  
 6 Unconstrained, it was rarely required for the Constrained scenario, emphasising why, for  
 7 ease, this latter mounting arrangement is specified in the Standard tests. Despite careful  
 8 headform positioning, the ball trajectory out of the cannon was inevitably somewhat  
 9 variable. A second high-frame-rate camera, a Photron FastCam Ultima APX mono  
 10 operating at 10 kHz, was therefore positioned directly above the impact location, as also  
 11 shown in Figure 3 (Camera 2), to enable an assessment of the impact location relative to  
 12 the  $yz$ -plane, but not used in measurements. By quickly interrogating the two camera

1 recordings immediately after impact, those for which the ball impact location was  
2 substantially different ( $\geq 5$  mm) to that desired were rejected and a repeated measurement  
3 performed. In practice, this impact-position criterion amounted to a repeat of between 1  
4 in 5 or 6 impacts which was considered a reasonable compromise between maximising  
5 data quality and minimising experiment-completion time. Similarly, variation in ball  
6 impact speed with respect to the required nominal was immediately estimated using a  
7 pair of timing light gates separated by a distance of 200 mm with the closest gate  
8 positioned 500 mm from the headform. With respect to this variation, trials were rejected  
9 if they deviated by more than  $\pm 2$  m/s from the target impact speed. This deviation led to a  
10 similar trial rejection rate to that produced by variations in impact location. It should be  
11 noted that actual impact speed values were determined from video data post-processing  
12 rather than from the light gates.



13  
14 Figure 4. Key frames from a typical (Constrained, top, 28 m/s) trial; (a) immediately  
15 prior to initial contact, (b) initial contact, (c) maximum ball deformation and (d)  
16 immediately following the end of contact.

17

## 1 Data post-processing

2  
3 A MATLAB script was written and used to enable efficient post-processing of the video  
4 data. The script employed edge detection techniques to identify the headform and the ball  
5 in each frame. The frame of initial contact (Figure 4b) was the frame at which the  
6 headform edge and ball leading edge first coincided while the frame immediately  
7 following the end of contact (Figure 4d) was that in which the two edges separated.  
8 Conversion from pixels to mm was performed by using the un-deformed ball diameter –  
9 determined using a digital Vernier caliper – at various positions across the image and  
10 found to be consistent. Ball deformation was determined on a frame-by-frame basis by  
11 determining the straight-line distance between the ball leading and trailing edges in the  $z$ -  
12 axis (as defined in Figure 1a) with deformation clearly at a maximum when this distance  
13 was at a minimum (Figure 4c). By further determining the positions of the three high-  
14 contrast circular target markers attached to the headform, as can be seen in Figure 4, the  
15 headform positioning, and any variability therein, prior to each impact could be verified.  
16 Accelerometer data were processed using Microsoft Excel and MATLAB with the  
17 previously set out sensitivities applied therein to convert from V to SI units. Analyses of  
18 the data sampled at 50 MHz revealed that no meaningful signal artefacts would be lost by  
19 down-sampling by a factor of 100. Given the accelerometer and video camera data  
20 capture were triggered using the same signal from the timing gates, corresponding  
21 datasets from each could be and were synchronised with respect to one another. The  
22 timestamp of the frames at the beginning and end of ball/headform contact, identified  
23 from high-frame-rate video data, were used to enable interrogation of the accelerometer  
24 data over that same contact period. The timestamp of the frame of maximum ball

1 deformation was used to determine the “loading” phase, defined as the period during  
2 ball/headform contact leading up to maximum ball deformation, and the “unloading”  
3 phase, defined as the contact period immediately following maximum ball deformation  
4 until ball/headform separation.

5 Accelerations in the  $x$ -,  $y$ - and  $z$ -directions were time-domain integrated using a  
6 trapezoidal rule to yield equivalent velocity components; initial non-zero offsets were  
7 eliminated by subtracting a mean value for the period of samples one video frame prior to  
8 impact so as to de-trend the outputs. Resultant speed traces were interrogated and, using  
9 all five trials for a given impact speed and location combination, a mean speed trace was  
10 derived (by taking the mean of the five values at each time sample). Interpretation of the  
11 video data, as indicated in Figure 4, enabled the frames (and therefore the times) at the  
12 start of contact, maximum ball deformation and end of contact, to be determined. The  
13 mean speed traces then being possible to be analysed in terms of the loading and  
14 unloading phases. Sensitivity analyses were undertaken to determine the impact of  
15 selecting a time based on the previous or subsequent frame at all three points of interest  
16 with differences in mean speed on the order of 1% maximum typically observed. During  
17 the processing of the accelerometer data, it was found that for one trial (Constrained,  
18 Bottom, 25-m/s) accelerometer data had not been recorded correctly and this trial was  
19 therefore discarded.

20 Consideration was given to normalisation, for example by deploying a regression  
21 analysis, of the results to attempt to account for variation in actual (as opposed to  
22 nominal) ball impact speed. Challenges with adopting such an approach, however,  
23 include not having a model to be able to confidently predict what the underlying

1 relationship should be for the regression analysis. Furthermore, normalisation to a  
2 singular value of impact speed would have clearly rendered tests for statistical  
3 significance of results impossible.

4 Where statistical significance tests have been completed, these were conducted using  
5 IBM SPSS version 22. Given that only two scenarios, Constrained and Unconstrained,  
6 are under consideration, multiple comparison tests are not relevant. Accordingly,  
7 following confirmation of normal distribution and homogeneity of variance using  
8 Shapiro-Wilk and Levene's tests, respectively, independent t-tests were used to identify  
9 differences between the two scenarios for the various impact conditions. Initially, the  
10 alpha level was set to 0.05. However, to adjust for family-wise errors that may occur due  
11 to the 45 independent t-tests, a Holm-Bonferroni correction was subsequently applied  
12 reducing the alpha level to 0.0011.

## 13 Results

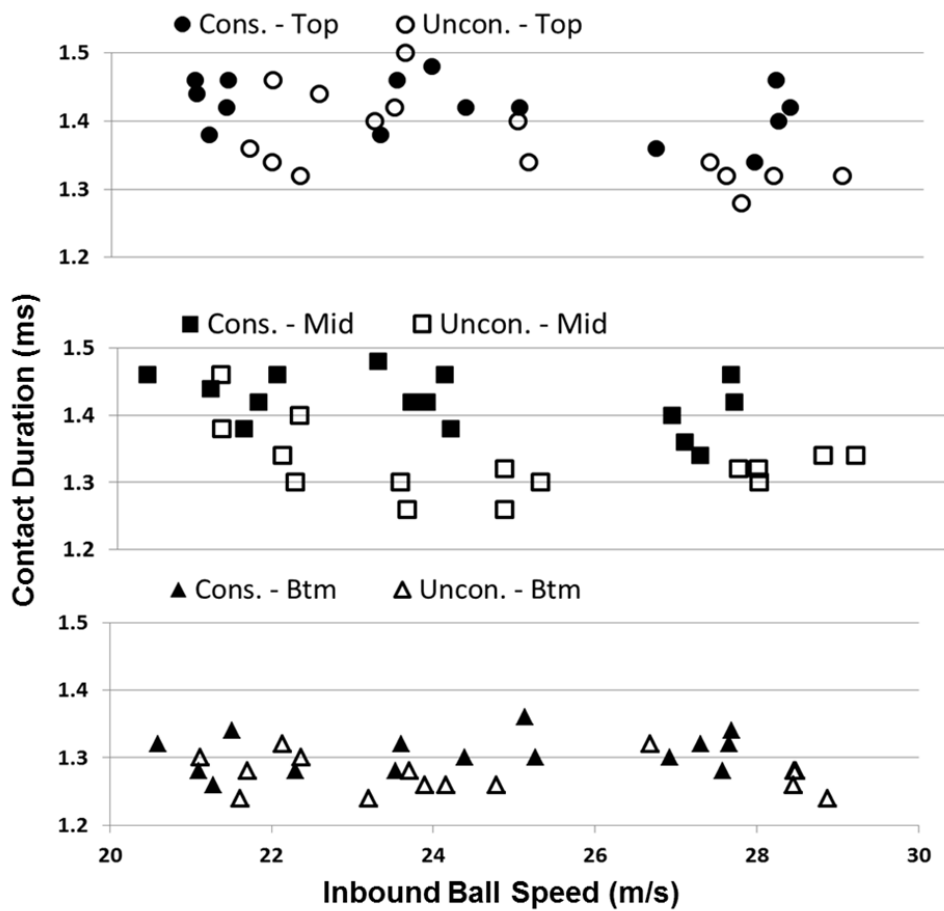
### 14 Contact duration

15  
16 The contact durations observed from high-frame-rate video recordings show no clear  
17 separation between equivalent speed/location impacts for Constrained ("Cons.-xxx") vs.  
18 Unconstrained ("Uncon.-xxx") scenarios as can be seen in Figure 5 and Table 1. It is  
19 evident from independent t-tests conducted that the only statistically significant  
20 difference between the Constrained and Unconstrained conditions occurs at the Middle-  
21 25-m/s condition ( $P = 0.0001$ ). While, upon initial inspection of these results, it may  
22 appear that increasing inbound ball speed leads to slightly shorter contact durations,  
23 standard deviations are significant relative to the differences between the means, as can  
24 be observed in Table 1. From the distribution of the data points in Figure 5, it is also



1 evident that variation in contact duration is generally reduced as the inbound speed  
2 increases, with the least variation at 28-m/s nominal ball speed. It is further evident from  
3 the standard deviations that this reduction in contact time is the case for the majority of  
4 impact location/mounting arrangement combinations, with the exception of Constrained-  
5 Top and Unconstrained-Bottom.

6 The largest variation within a specific testing condition was for Unconstrained-Top, 22-  
7 m/s nominal ball speed. Limited variation in contact duration may be the result of the  
8 limited resolution to identify the key frames where contact begins and ends. Contact  
9 duration variation determination is also a function of variation in the impact location  
10 laterally which, due to the curved nature of the headform, may lead to small but  
11 measureable differences in determined contact duration. At the bottom impact location,  
12 however, the headform surface profile is much flatter and deviations laterally should not  
13 result in as significant a change in the determined contact duration than may be the case  
14 at the top and middle locations. The largest difference between the Constrained and  
15 Unconstrained scenarios occurred at the Middle, 25-m/s nominal ball speed, where the  
16 Constrained scenario displayed a mean contact duration 0.14 ms greater than that of the  
17 Unconstrained equivalent.



1

2 Figure 5. Comparison between contact durations and measured inbound ball speeds.

3 Table 1. Means and standard deviations for contact durations and measured inbound ball  
 4 speeds.

Impact Location	Constrained				Unconstrained			
	Ball Speed (m/s)		Contact Duration (ms)		Ball Speed (m/s)		Contact Duration (ms)	
	Mean	SD	Mean	SD	Mean	SD	Mean	SD
Top	27.9	0.6	1.40	0.04	27.6	0.6	1.32	0.02
	24.2	0.5	1.43	0.03	24.0	0.8	1.41	0.05
	21.3	0.2	1.43	0.03	22.0	0.4	1.38	0.06
Middle	27.1	0.3	1.40	0.04	28.1	0.8	1.32	0.01
	23.6	0.4	1.43	0.03	24.0	0.8	1.29	0.02
	21.3	0.6	1.43	0.03	21.8	0.4	1.38	0.05
Bottom	27.4	0.3	1.31	0.02	28.1	0.6	1.28	0.03
	24.2	0.6	1.31	0.03	24.0	0.7	1.26	0.01
	21.5	0.6	1.30	0.03	21.6	0.5	1.29	0.03

5

6

## 1 Ball deformation

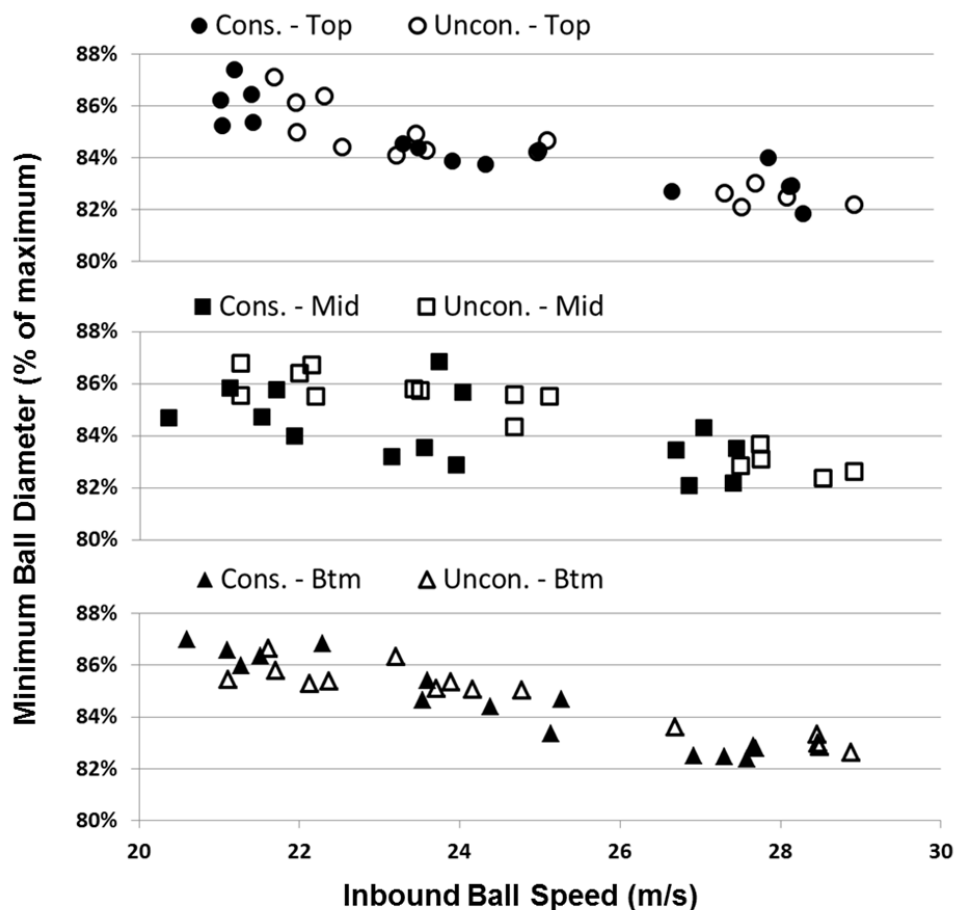
2  
 3 When comparing the duration from initial contact to maximum ball deformation, as  
 4 shown in Table 2, it appears that this contact time is slightly greater for the Constrained  
 5 scenario than for the Unconstrained. However, when the standard deviations are also  
 6 considered, it is apparent that, generally, there is little difference between equivalent  
 7 scenarios for the two mounting arrangements. This lack of any statistically significant  
 8 difference is confirmed by independent t-tests between the two scenarios for all impact  
 9 conditions.

10 Table 2. Means and standard deviations for times from initial contact to maximum ball  
 11 deformation (inbound ball speed equivalents included again for completeness).

	Constrained				Unconstrained			
	Ball Speed (m/s)		Time to Max Ball Def. (ms)		Ball Speed (m/s)		Time to Max Ball Def. (ms)	
	Mean	SD	Mean	SD	Mean	SD	Mean	SD
<b>Top</b>	27.9	0.6	0.57	0.03	27.6	0.6	0.55	0.02
	24.2	0.5	0.62	0.04	24.0	0.8	0.55	0.03
	21.3	0.2	0.62	0.03	22.0	0.4	0.55	0.04
<b>Middle</b>	27.1	0.3	0.57	0.02	28.1	0.8	0.55	0.02
	23.6	0.4	0.59	0.04	24.0	0.8	0.54	0.02
	21.3	0.6	0.54	0.06	21.8	0.4	0.54	0.03
<b>Bottom</b>	27.4	0.3	0.56	0.07	28.1	0.6	0.50	0.01
	24.2	0.6	0.61	0.04	24.0	0.7	0.53	0.03
	21.5	0.6	0.56	0.07	21.6	0.5	0.53	0.04

12  
 13 As should be expected and as can be clearly observed in Figure 6, for both the  
 14 Constrained and Unconstrained mounting arrangement scenarios, the maximum ball  
 15 deformation increased with increasing inbound ball speed. From the distribution of these  
 16 data, it would appear that, despite relatively large variations possibly due to the limited  
 17 spatial pixel resolution of the video data from which they are derived, generally greater  
 18 ball deformation occurs in the Constrained scenarios compared to their Unconstrained  
 19 equivalents. Table 3 shows the means and standard deviations for minimum ball

1 diameters, i.e. at maximum ball deformation, against mean ball speeds and, with the  
 2 exception of 28-m/s impacts at the Middle impact location, the Constrained scenario  
 3 displayed slightly greater mean ball deformation at all combinations of impact location  
 4 and nominal speed. Despite the differences between the Constrained and the  
 5 Unconstrained trials presented in Figure 6, Table 3 confirms that these differences are  
 6 generally relatively small with relatively high standard deviations. Independent t-tests  
 7 identified the differences in ball deformations between the Constrained and  
 8 Unconstrained scenarios to be only statistically significant at the Bottom, 28-m/s  
 9 combination ( $P = 0.001$ ).



10

11 Figure 6. Comparison between minimum ball diameters and measured inbound ball  
 12 speeds.

1 Table 3. Means and standard deviations for minimum ball diameters while in contact with  
 2 the headform (inbound ball speed equivalents included again for completeness).

Impact Location	Constrained				Unconstrained			
	Impact Speed (m/s)		Min Ball Diameter (Relative to Max)		Impact Speed (m/s)		Min Ball Diameter (Relative to Max)	
	Mean	SD	Mean	SD	Mean	SD	Mean	SD
Top	27.9	0.6	82.8%	0.6%	27.6	0.6	83.6%	0.4%
	24.2	0.5	83.9%	0.4%	24.0	0.8	84.7%	0.6%
	21.3	0.2	85.9%	0.7%	22.0	0.4	86.1%	1.3%
Middle	27.1	0.3	83.2%	1.0%	28.1	0.8	82.8%	0.9%
	23.6	0.4	84.8%	1.1%	24.0	0.8	86.2%	0.9%
	21.3	0.6	85.1%	0.8%	21.8	0.4	86.1%	0.9%
Bottom	27.4	0.3	82.5%	0.3%	28.1	0.6	83.5%	0.3%
	24.2	0.6	84.7%	0.5%	24.0	0.7	85.3%	1.0%
	21.5	0.6	86.2%	0.9%	21.6	0.5	86.4%	0.5%

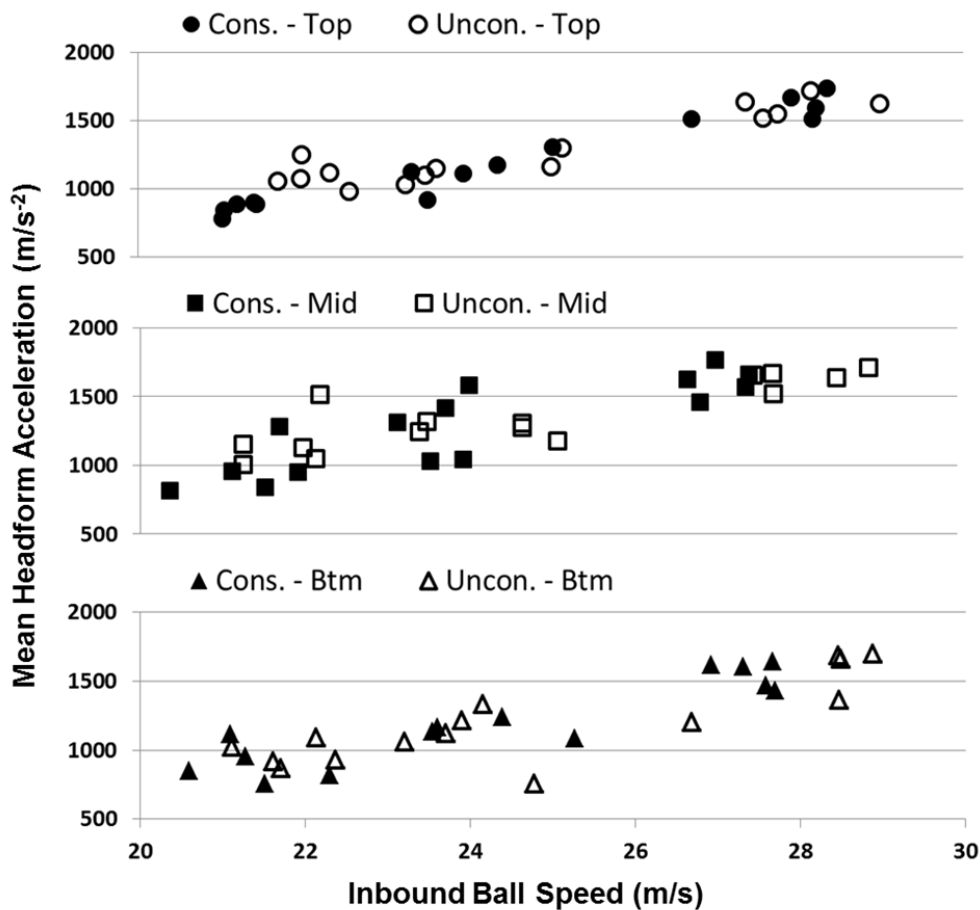
3

#### 4 Headform acceleration

5

6 In all equivalent mounting arrangement scenarios, mean resultant acceleration during the  
 7 loading phase was found to increase with inbound ball speed, as can be readily observed  
 8 in Figure 7. The Constrained and Unconstrained groups of data points are distributed  
 9 similarly and over a similar range, particularly for the 28-m/s ball speed impacts. Such  
 10 characteristics suggest similar measurement variabilities and uncertainties for each  
 11 arrangement and that, therefore, differences between the derived measures are not a  
 12 function of different aspects of the two sets of experiments, for example substantial  
 13 differences in the projection of the ball for the Constrained and Unconstrained setups. At  
 14 22-m/s, there appears to be greater separation between the Constrained and  
 15 Unconstrained data points, indicating that the headform displayed slightly greater mean  
 16 accelerations in the Unconstrained scenarios. These observations are reflected in the  
 17 mean average resultant acceleration statistics included in Table 4, where there are  
 18 generally no distinct differences between the Constrained and Unconstrained equivalent

1 scenarios particularly at 28- and 25-m/s ball speed. There are, however, differences at 22-  
 2 m/s as the Unconstrained data exhibit a higher mean resultant acceleration at both the  
 3 Top and Middle impact locations. Figure 7 suggests that, where variability was high,  
 4 these differences in mean resultant acceleration are due to genuine distribution of data  
 5 points, as opposed to outliers skewing the statistics. The only statistically significant  
 6 difference between the scenarios was found at an impact speed of 22-m/s at the Top  
 7 impact location ( $P < 0.001$ ).



8

9 Figure 7. Comparison between mean resultant accelerations during the loading phase and  
 10 measured inbound ball speeds.

11

12

- 1 Table 4. Means and standard deviations of mean resultant accelerations during the
- 2 loading phase (inbound ball speed equivalents included again for completeness).

Impact Location	Constrained				Unconstrained			
	Impact Speed (m/s)		Mean Resultant Acceleration (m/s <sup>2</sup> )		Impact Speed (m/s)		Mean Resultant Acceleration (m/s <sup>2</sup> )	
	Mean	SD	Mean	SD	Mean	SD	Mean	SD
Top	27.9	0.6	1606	88	27.6	0.6	1610	72
	24.2	0.5	1127	126	24.0	0.8	1149	87
	21.3	0.2	860	45	22.0	0.4	1096	88
Middle	27.1	0.3	1615	101	28.1	0.8	1638	63