



HSE Occupational Health & Safety and Environmental Protection unit



Measurement of heat flux in multi-layer insulated helium cryostats after loss of insulating vacuum

C1Or2A-07 – Applications: Safety and Instrumentation

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Outline

Motivation

- Experimental setup
- > Heat transfer mechanism
- Results & Discussion
- Summary & Outlook

PICARD <u>Pressure Increase in Cryostats and Analysis of Relief Devices</u>







Motivation

Profound understanding of the dimensioning process of Pressure Relief Devices (PRD)

- Thermodynamic process tailored to the application;
- Uniform & detailed understanding of heat transfer mechanisms.
- Compare with existing data for continuous improvement of the 'state-of-the-art'
- Emphasis on heat flux in multi-layer insulated (MLI) helium cryostats after loss of insulating vacuum (LIV)







Experimental setup







Characteristics

Туре	N _{Layer}	Thickness (µm)		Perforation holes (mm)	
		δ_{R}	$\delta_{ m S}$	ф	Grid
1	12	6	55	2	50
2	1	18	-	6	200
3	10	12	55	4	150



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Thermal bridges warm-cold layers avoided using aluminum adhesive

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Heat transfer mechanism







Heat transfer mechanism



<u>1D heat transfer equation for cryogenic wall temperature</u> $(T_{w,i} \& T_{w,o})$

Deposition heat flux Thermal radiation : Enthalpy balance – $f(\dot{M}_{in}, p_v)$

: Stefan-Boltzmann equation

Thermal conduction MLI : <u>Fourier</u> equation (radial)

Thermal conduction wall : Fourier equation

Heat transfer in helium : <u>Convective heat transfer</u>

Note:

- Vacuum vessel: Convective heat transfer neglected due to low Grashof numbers
 - Deposition: Solid air enthalpies included (ideal mixture of N₂, O₂, Ar, and water)













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- Plot shape (Vacuum space):
 - Influence on the flow resistance of the MLI blanket 1)



- Plot shape (Helium):
- $\dot{q}_{\rm He}$ small due to film boiling 1)







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- Plot shape (Vacuum space):
 - 1) Influence on the flow resistance of the MLI blanket
 - 2) Increase of heat flux to peak due to $p_v \approx atm$

Plot shape (Helium):

- 1) \dot{q}_{He} small due to film boiling
- 2) Peak due to property data in vicinity of *p*_{Crit}







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Plot shape (Vacuum space):

- 1) Influence on the flow resistance of the MLI blanket
- 2) Increase of heat flux to peak due to $p_v \approx atm$
- 3) First opening of PRD

Plot shape (Helium):

- 1) \dot{q}_{He} small due to film boiling
- 2) Peak due to property data in vicinity of p_{Crit}
- 3) Free convection
 First opening of the PRD –
 Heat flux relevant for dimensioning

 From ~ 1.2 0.7 W/cm²







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Plot shape (Vacuum Space):

- 1) Influence on the flow resistance of the MLI blanket
- 2) Increase of heat flux to peak due to $p_v \approx atm$
- 3) First opening of PRV
- 4) Heat flux limited by cryopumping effect

Plot shape (Helium):

- 1) \dot{q}_{He} small due to film boiling
- 2) Peak due to property data in vicinity of p_{Crit}
- 3) Free convection
 First opening of the PRD –
 Heat flux relevant for dimensioning
 From ~ 1.2 0.7 W/cm²
- 4) Heat flux quasi-independent of the MLI type





Results – Exp. 3 vs Exp. 4





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100%

area

1250

550

Results

Comparison with literature

NLayers	Reference	Heat flux (W/cm ²)	
0	Lehmann & Zahn [12]	3.8	
	PICARD	1.4	
1	Lehmann & Zahn [12]	2.0	
	PICARD	1.2	
10	Lehmann & Zahn [12]	0.6	
	PICARD	0.7	
12	Lehmann & Zahn [12]	0.59*	
	PICARD	1.0	
24	Lehmann & Zahn [12]	0.38*	
	PICARD	0.7	

*extrapolated



- 12 layer blankets higher values
- Bare & 1 Layer lower values



Summary and Outlook

Summary

- Experiments performed at PICARD with MLI
- Effect of multi-layer insulated helium cryostats was measured and evaluated
- Influence on manufacturing characteristics of the MLI demonstrated (e.g. perforation holes)
- Relevant heat flux for dimensioning of PRD discussed
- Results differ from literature higher heat flux for Type 1 MLI

Outlook

- Further experimental investigations on types of MLI
- Investigate the possibility of an 'equivalent' MLI resistance in the dynamic model
- Evaluation of model uncertainty (Bayesian approach)

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Thank you for your attention

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Spare Slides





Heat transfer mechanism - MLI



Density: 10 layers / cm

 $\dot{q}_{Cond,n}$: Residual gas & solid thermal conduction (*n* ; *n*+1)

- Total thermal resistance derived from [6], including:
 - Gaseous air, spacer and the reflective screens

Radiated heat considering *N* reflective MLI layers as grey emitters. Emissivity values:

- ϵ_v = 0.8 vacuum vessel (oxidized SS)
- ϵ_{Cr} = 0.07 helium vessel (electro-polished SS)
- ϵ_{MLI} = 0.04 reflector (electro-polished AI)

ġ_{Dep}∶

Measured based on mass flow of venting air and rise in vacuum pressure [7]





Heat transfer mechanism - Helium



- $\dot{q}_{\rm He}$: Heat transfer coefficient $\alpha_{\rm He}$ depends on thermodynamic state and fluid phase
 - Correlations <u>subcritical</u> state:
 - $A_{\rm Cr}$ in contact with liquid $\alpha_{\rm He}$ pool boiling [8,9] $A_{\rm Cr}$ in contact with gas $\alpha_{\rm He}$ free convection [10]
 - Correlations <u>supercritical</u> state: α_{He} - free convection [11]





Formulas

- Cryo wall temperatures:
- Stefan-Boltzmann:
- Fourier (cryo wall):

Fourier (MLI):

 $\dot{q}_{\text{Cond,n}} = \frac{(r_{\text{n}+1} - r_{\text{n}}) \cdot (T_{\text{n}+1} - T_{\text{n}})}{r_{\text{n}} \cdot \left(R_{\text{g}} + \frac{R_{\text{s}} \cdot R_{\text{s}}'}{R_{\text{s}} + R_{\text{s}}'} + R_{\text{r}}\right) \cdot \ln\left(\frac{r_{\text{n}+1}}{r_{\text{n}}}\right)}$

 $\dot{q}_{\rm He} = \alpha_{\rm He} \cdot (T_{\rm W,i} - T_{\rm He})$

Convective heat flux to He:

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Deposition heat flux:

$$\dot{q}_{\text{Dep}} = rac{\dot{M}_{ ext{Dep}}}{A_{ ext{Cr}}} \cdot \left(h_{ ext{air}}\left(p_{ ext{amb}}, T_{ ext{amb}}, \varphi_{ ext{amb}}
ight) - h_{ ext{air}}\left(p_{ ext{V}}, T_{ ext{W}_{ ext{o}}}, \varphi_{ ext{amb}}
ight)
ight)$$

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 $\begin{aligned} \frac{\mathrm{d}T_{\mathrm{W,o}}}{\mathrm{d}t} &= \frac{A_{\mathrm{Cr}}}{c_{\mathrm{Cr}} \cdot M_{\mathrm{Cr}}} \cdot \left(\dot{q}_{\mathrm{Dep}} + \dot{q}_{\mathrm{Rad}} + \dot{q}_{\mathrm{Con}} - \dot{q}_{\lambda,\mathrm{W}}\right) \quad ; \qquad \frac{\mathrm{d}T_{\mathrm{W,i}}}{\mathrm{d}t} &= \frac{A_{\mathrm{Cr}}}{c_{\mathrm{Cr}} \cdot M_{\mathrm{Cr}}} \cdot \left(\dot{q}_{\lambda,\mathrm{W}} - \dot{q}_{\mathrm{He}}\right) \\ \dot{q}_{\mathrm{Rad}} &= \sigma \cdot \left(T_{\mathrm{V}}^{4} - T_{\mathrm{W}}^{4}\right) \cdot \left(\left(\frac{1}{\epsilon_{\mathrm{Cr}}} + \frac{1}{\epsilon_{\mathrm{MLI}}} - 1\right) + \left(N - 1\right) \cdot \left(\frac{2}{\epsilon_{\mathrm{MLI}}} - 1\right) + \left(\frac{1}{\epsilon_{\mathrm{MLI}}} + \frac{1}{\epsilon_{\mathrm{V}}} - 1\right)\right)^{-1} \\ \dot{q}_{\lambda,\mathrm{W}} &= \frac{\lambda_{\mathrm{Cr}}}{s_{\mathrm{Cr}}} \cdot \left(T_{\mathrm{W,o}} - T_{\mathrm{W,i}}\right) \end{aligned}$

Heat flux plot





