

Thermo-economic and environmental analysis of integrating renewable energy sources in a district heating and cooling network

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Abstract

This paper presents the technical, environmental and economic evaluation of integrating various combinations of renewable energy sources-based systems in the expansion of a district heating and cooling network of a Technology Park near Barcelona in Spain. At present, a Combined Heat and Power plant running on fossil fuels serves the heating, cooling and electricity demand of the Park. However, this energy demand is expected to increase substantially in the coming years. EnergyPRO software was used to model the energy demand growth till 2030. Validation of the software application was done by making a base model using real plant data from the year 2014. The software was then used to project the energy supply based on three 15-year scenarios, having different combinations of renewable energy technologies, from 2016 until 2030. Primary energy consumption, CO₂ emissions and the Net Present Value obtained in each scenario were used to decide the best combinations of renewable energy sources. The results of the study showed that presently, biomass boilers combined with absorption chillers and supported with solar thermal cooling, are the most competitive technologies in comparison to ground source heat pumps for large DHC networks. This is mainly because of the lower primary energy consumption (624,380 MWh/year in 2030 vs. 665,367 MWh/year), higher Net Present Value (NPV) (222 million € vs. 178 million €), and lower CO₂ emissions (107,753 tons/year in 2030 vs. 111,166 tons/year) obtained as a result of the simulations.

Keywords: District heating and cooling, renewable energy integration, energy efficiency, feasibility study, techno economic evaluation

1 Nomenclature

2 Greek Letters

η_{el} Electrical efficiency (%)

η_{th} Thermal efficiency (%)

Subscripts

c Cooling

h Heating

th Thermal

el Electric

3 Acronyms

BAU	Business As Usual	GSHPs	Ground Source Heat Pumps
CAPEX	Capital Expenditure	IT	Information Technology
CHP	Combined Heat and Power	KPIs	Key Performance Indicators
CO ₂	Carbon dioxide	NZEBs	Net Zero Energy Buildings
COP	Coefficient of Performance	P & L	Profit and Loss
DH	District Heating	PEF	Primary Energy Factor
DHC	District Heating and Cooling	PTCs	Parabolic Trough Collectors
EBITDA	Earnings Before Interests, Taxes Depreciation and Amortization	RES	Renewable Energy Sources
EU	European Union	SCBC	Solar cooling and biomass cooling
GHG	Green House Gas	TES	Thermal Energy Storage

4 1. Introduction

5 The increase in price of fossil fuels, their rapid depletion and environmental impact have
6 accelerated research work in Renewable Energy Sources (RES) and thus RES are playing an
7 important role in future energy systems. Moreover, fossil fuels are a major contribution to
8 greenhouse gas (GHG) emissions, due to which several countries and international bodies are
9 setting immediate goals to combat climate change. The European Union (EU) for instance, has

1 set the ambitious target of having at least 27 % share for renewable energy in its total energy mix
2 by 2030, as per the 2030 climate and energy framework [1].

3 Along with an increased push in using renewables, it must be mentioned that buildings account
4 for 40% of energy usage and CO₂ emissions in the EU [2]. To mitigate the contribution of energy
5 sector to climate change, several policies have been designed and implemented, which supports
6 the development of high-efficiency Combined Heat and Power (CHP) plants in Europe [3]. CHP
7 plants are not only energy efficient, but may also curb CO₂ emissions if powered by RES based
8 systems. Thus, to help the EU meet its ambitious targets, integration of RES in District Heating
9 and Cooling (DHC) networks appears to be an excellent solution.

10 Several studies have been carried out for evaluating the technical, economic and environmental
11 feasibility of integrating RES in DHC networks. Dagdougui et al. [4] developed a dynamic
12 model to integrate different RES and a storage device to satisfy the thermal and electric demand
13 of a “Green Building”. Wang et al. [5] used a modelling and optimizing technique for
14 developing CHP based District Heating (DH) system, with a solar thermal plant and Thermal
15 Energy Storage (TES) system. Results of the analysis proved that the model is suitable for
16 planning and running CHP-DH systems economically. A simulation was then run with higher
17 proportion of RES and a larger TES, indicating that the TES is utilized more with higher share of
18 RES and a fluctuating load of the CHP-DH system. Østergaard [6] , using the EnergyPRO
19 software, developed a 100% renewable energy scenario for the Danish city of Aalborg and then
20 compared the overall impact on the system of various energy storage systems (DH storage,
21 biogas storage and electricity storage). Nielsen and Moller [7] carried out a study for Denmark
22 where they modelled the integration of solar thermal collectors into Net Zero Energy Buildings
23 (NZEBS) connected to district heating (DH) networks, so that the collectors could satisfy part of
24 the heating demand. The results show that the NZEBs experienced a net decrease in heat supply
25 from the network (CHP units and boilers), and hence decrease in burning of fuels. Streckiene et
26 al. [8] modelled the optimum CHP plant with thermal energy storage that would be most
27 suitable to take advantage of the day ahead German electricity market. The results of the study
28 showed that for a CHP plant supplying 30,000 MWh_{th,h} thermal energy annually, a 4MW_e
29 capacity plant with a thermal store would be most technically and economically feasible for
30 participating in the spot market. For district heating systems in Lithuania, Lund et al. [9] did a

1 study where they analysed the replacement of old boilers burning fossil fuels with CHP units
2 running on renewable sources such as wood. Results showed that this would considerably
3 decrease fossil fuel consumption by 50-70% and CO₂ emissions by 50% or 70% (depending
4 upon the scenario), and also bring down the district heating prices in small towns to the level of
5 those in urban areas. Thermo economic analysis have been performed of district heating and
6 cooling systems connected to ground source heat pumps (GSHPs) in [10], [11], [12] and [13].
7 Carli et al. [11] showed that in comparison to traditional heat pumps, using GSHPs can reduce
8 primary energy consumption between 50 and 80%. Similar studies, utilizing exergy methods,
9 have been carried out for geothermal district heating systems in [14] and [15]. An interesting
10 thermo economic optimization was performed by Baghernejad et al. [16] of a solar assisted
11 trigeneration (cooling, heating and electricity) system. An optimization model was developed by
12 Buoro et al. [17] for a solar thermal system (with heat storage) integrated with a CHP plant. The
13 study showed that the most economic and environmental friendly scenario is one where the solar
14 field, thermal storage and district heating network operate together. Torchio [18] did an analysis
15 to compare district heating CHP with distributed generation (on-site) CHP. Of the three
16 technologies used for the analysis, fuel cells, micro turbines and internal combustion engines, the
17 results showed that the lowest amount of carbon dioxide emissions was obtained when fuel cells
18 are used as the CHP units in a district heating network. Soltero et al. [19] did a study to replace
19 existing individual heating systems and old coal and nuclear power stations in the Spanish city of
20 Burgos with CHP district heating. Results showed that with this replacement, annually 4 million
21 tons of CO₂ emissions could be avoided and profits of 300 million € could be made. Kazagic et
22 al. [20] did a feasibility study for the city of Visoko where they proposed a renewable energy
23 based DH system to replace the current coal fired CHP plants. Although the investment costs
24 were higher for the renewable energy based system, the fuel costs and CO₂ emissions were 58%
25 and 46% less respectively than that of the coal based system. Rämä et al. [21] did a feasibility
26 study for the city of Helsinki in Finland where they modelled an increased the share of renewable
27 heat sources (solar thermal collectors and water-to-water heat pumps) in the existing DH system.
28 Results showed that heat pumps are a better option when compared to solar collectors, not only
29 in terms of environmental emissions but also cost effectiveness.

30 Despite a large number of publications available on the topic, there is lack of a complete
31 feasibility study i.e., technical, economic and environmental analysis on the establishment of

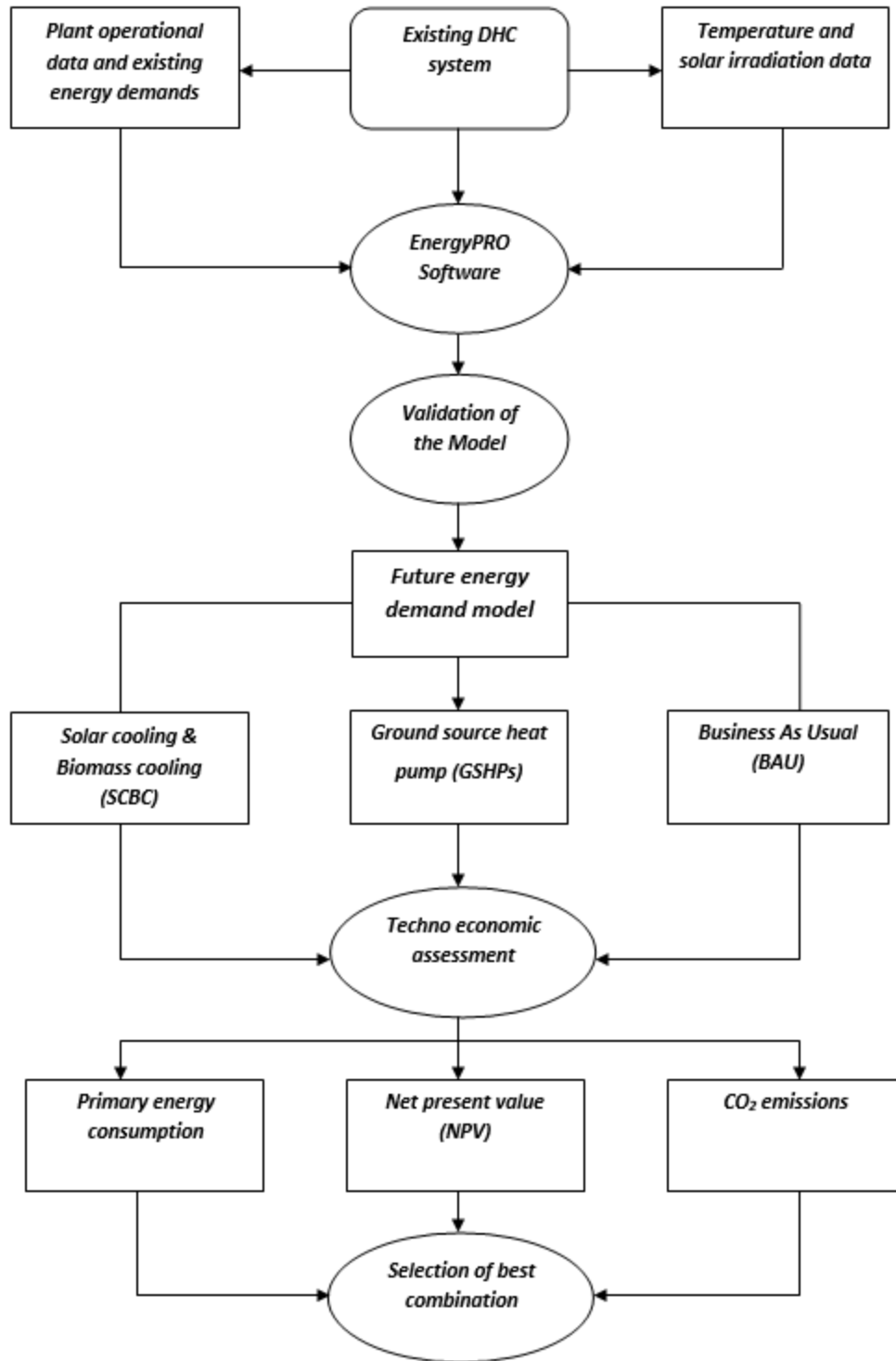
1 RES based DHC in a Mediterranean climatic zone in Spain. This may be perhaps because there
2 is generally a lack of regulations for DHC networks in Spain [22]. It is challenging to attract
3 investors for setting up new DHC networks because of the long term amortization, lack of
4 funding from banks and low public investment. Moreover, for cultural and societal reasons,
5 cooling and heating customers prefer individual domestic systems in comparison to collective
6 DHC systems.

7 This article aims to form a basis for promoting DHC networks based on RES. It presents the
8 technical, environmental and economic evaluation of integrating RES for the expansion of an
9 existing DHC system in Parc de l' Alba, a Technology Park near Barcelona in Spain. The reason
10 for choosing this specific location is the fact that it lies in the region of Catalonia in north east
11 Spain, next to the Mediterranean Sea and bordering France at the Pyrenees mountains. Thus, the
12 region has a variety of different climatic zones, making it a suitable location to establish DHC
13 networks.

14 Currently, a CHP plant, titled ST-4, serves the demand of cooling, heating and electricity of the
15 non-residential consumers but there is a forecast of increased demand over the years. An energy
16 model is created initially and validated, after which the demand growth is modeled. Three 15-
17 years projection scenarios are created incorporating different combinations of RES. The best
18 combinations of RES are decided based on the CO₂ emissions, primary energy consumption and
19 Net Present Value (NPV), which have been considered as the Key Performance Indicators (KPIs).

20 **2. Material and Methods**

21 The detailed layout of the methodology is shown in Figure 1. The very first step, as explained
22 previously, was creation of base model in 2015 to ensure that future analysis of scenarios would
23 be accurate. Real data from the plant (for year 2014) were obtained and provided to EnergyPRO
24 [23] software, which is the primary software tool used for calculations in this study. Based on
25 several inputs including demand profile, weather data, tariffs, efficiencies and capacities of
26 energy conversion units, EnergyPRO gives a comprehensive output on economics, emissions and
27 operational strategy.



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Figure 1: Methodology of the feasibility study

3 The initial inputs to the base model were Time Series of hourly temperature data and solar
 4 irradiation of Barcelona. These climate time series were obtained from an online database. To

1 generate the electricity market of Spain [24] , the following time series were provided as input:

- 2 • **Pool Price (TS1)** It denotes pool price for the year, continuously varying on hourly basis
3 between 0 €/MWh_e and 114 €/MWh_e for the year 2016
- 4 • **57€/MWh_e (TS3)** Time series with a constant value of 57 €/MWh_e in 2016. According to
5 Spanish legislation, it corresponds to the payment for system operation
- 6 • **Spanish Tariff Hours (STH)** It denotes fraction of electricity price dependent on
7 different time periods of the day when buying from the grid. Varying on hourly basis
8 between 7.0 €/MWh_e and 27.9 €/MWh_e for the year 2016

9 Three Time Series Functions were utilized to compute costs and revenue related to electricity
10 export and import. They are:

- 11 • **Price Electricity Imported Time Series (STHF)** Addition of two time Series, STH and
12 TS1, denoting the amount of money paid by ST-4 if electricity is purchased from the grid.
13 It is defined by equation (1)

$$STHF = STH + TS1 \quad (1)$$

- 14 • **Price Electricity Exported Time Series (Fu1)** Addition of two time series, TS3 and
15 TS1, denoting the amount of money ST-4 earns when electricity is exported to the grid.
16 It is defined by equation (2).

$$Fu1 = TS3 + TS1 \quad (2)$$

- 17 • **Exported Electricity Tax (Fu2)** It denotes the tax paid on 7% of the electricity exported
18 to the grid. It is defined by equation (3).

$$Fu2 = Fu1 * 0.07 \quad (3)$$

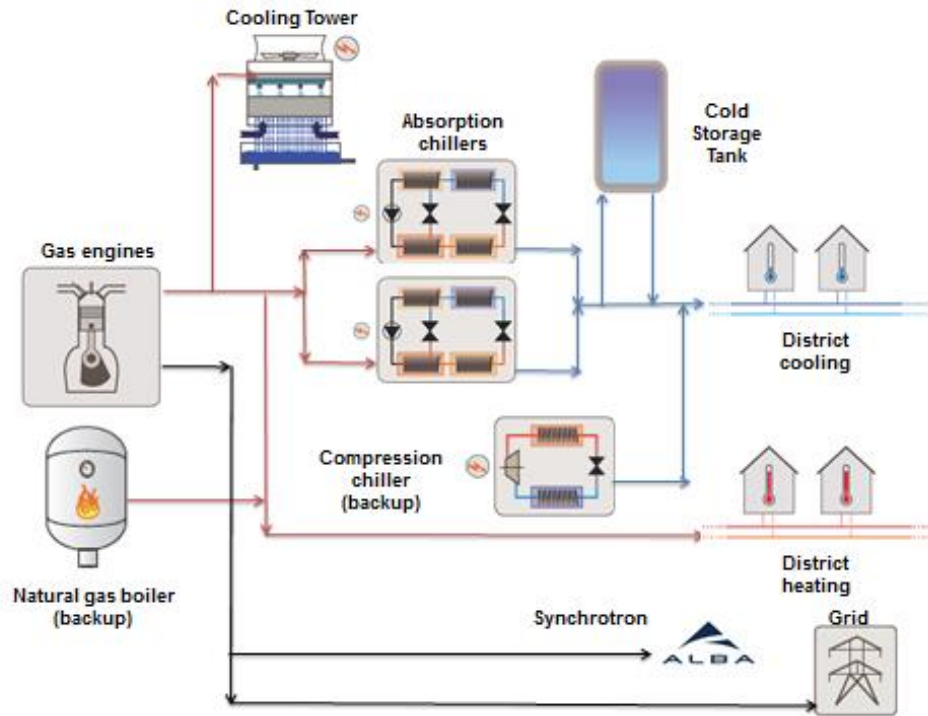
19 Currently, natural gas is burnt in the engines that produce electricity and heat. This heat is sent to
20 the district heating network and is fed to the two absorption chillers to produce cooling for the
21 district cooling network. Excess cooling is stored in the cold-water storage tank. The cooling
22 tower ensures removal of any excess heat from the engines. When electricity prices are low, the
23 engines are turned off. The natural gas boiler and compression chiller serves the heating and

1 cooling demand respectively; the cold-water storage tank is also discharged in case the
 2 compression chiller is not enough. Key details of all energy conversion units and cold storage
 3 tank at the ST-4 plant are shown in Table 1. The operating scheme of the plant is shown in
 4 Figure 2. It is assumed that the electrical and/or thermal energy conversion efficiencies of the
 5 different energy conversion units remains the same even at part load operation.

6 *Table 1: Details of ST-4 plant at Parc de l'Alba (2015)*

Unit type	Quantity	Specifications (each unit)	Comments
Cogeneration Engines	3	$3.28 \text{ MW}_{\text{th,h}}$; $3.35 \text{ MW}_{\text{e}}$; $\eta_{\text{el}} = 44.9\%$; $\eta_{\text{th}} = 41.1\%$	Turned on together whenever electricity spot markets are high
Single Effect Absorption Chiller	1	$3 \text{ MW}_{\text{th,c}}$; $\text{COP} = 0.7$	Driven by hot water from the engines at 90°C
Double Effect Absorption Chiller	1	$5 \text{ MW}_{\text{th,c}}$; $\text{COP} = 1.3$	Driven by exhaust gases of the engines at 398°C
Natural gas boiler	1	$5 \text{ MW}_{\text{th,h}}$; $\eta_{\text{th}} = 60\%$	Back-up
Compression Chiller	1	$5 \text{ MW}_{\text{th,c}}$; $\text{COP} = 5$	Back-up
Cold water storage tank	1	4000 m^3 ; $21 \text{ MWh}_{\text{th,c}}$	
Total plant capacity (excluding backup): $8.0 \text{ MW}_{\text{th,c}}$, $9.8 \text{ MW}_{\text{th,h}}$ and $10.1 \text{ MW}_{\text{e}}$			

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Figure 2: Current operational scheme of ST-4 plant at Parc de l'Alba

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The fuel used is natural gas, so it is input to the model with a calorific value of 10.64 kWh/m³[25]. In 2015, the ST-4 plant serves energy demands of the Synchrotron (particle accelerator) facilities and an office building (called “Plot 1” in the scope of this study for reasons of not disclosing the actual name of the client), details of which are shown in Table 2. It needs to be mentioned here that the different demand profiles for the various consumers are not considered for the ones in 2015, but have been taken into account for analysis of future scenarios.

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Table 2: Annual energy demands served by ST-4 in 2015

Type of demand	Consumer	Value
Cooling demand	Synchrotron	21,700 MWh _{th,c}
	Plot 1	651 MWh _{th,c}
Heating demand	Synchrotron	900 MWh _{th,h}
	Plot 1	530 MWh _{th,h}
Electricity demand	Synchrotron	20,400 MWh _e
	In-house consumption and losses	5,320 MWh _e

2

3 The plant generates income not only by selling heating and cooling, but also by selling electricity
4 to the Synchrotron and grid. The Appendix shows details of the revenues, along with the plant
5 expenses, as obtained from the Parc de l'Alba management [25].

6 After providing all the afore mentioned inputs, the simulation was run. The model optimizes
7 plant operation according to the electricity pool prices and electricity production costs. Data was
8 obtained for an entire year from the simulation in hourly basis, and validated against actual
9 results. Table 3 shows the comparison between the actual figures obtained from the plant and the
10 simulation results, for the year 2015 for the months of January, February and July-December. In
11 the months of March-June, the compression chiller had broken down and the resulting change in
12 plant operation was not taken into account by the software. Hence, comparison between
13 simulation and real data for these months is not presented.

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1 *Table 3: Comparison of actual plant operational data of Parc de l' Alba compared against results of the*
 2 *EnergyPRO simulation for selected months of year 2015 (all months of the year excluding March-June)*

Parameter	Actual data	Simulation result	Percentage difference
Cooling demand	20,500 MWh _{th,c}	22,400 MWh _{th,c}	-9%
Heating demand	1,430 MWh _{th,h}	1,430 MWh _{th,h}	0%
Electricity demand	25,700 MWh _e	25,700 MWh _e	0%
Electricity production by gas engines	35,600 MWh _e	36,900 MWh _e	-4%
Exported electricity	21,900 MWh _e	25,900 MWh _e	-18%
Imported electricity	12,100 MWh _e	16,400 MWh _e	-36%
Net electricity export	9,800 MWh _e	9,500 MWh _e	3%
Natural Gas Boiler Fuel consumption	1,950 MWh _{th,h}	1,500 MWh _{th,h}	24%
Natural Gas Boiler Heat production	1,170 MWh _{th,h}	890 MWh _{th,h}	24%
Duration of operation of gas engines	3,600 hours	3,700 hours	-3%

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 4 Although the simulation results of imported, exported and net exported electricity show a
 5 considerable difference from the actual data, it can be deduced that the software is reliable for
 6 estimation of future scenarios. This is because the electricity markets of Spain are extremely
 7 complex and cannot be very precisely modelled by EnergyPRO.

8 Next, EnergyPRO models were created for the years in which new demands were incorporated,
 9 namely 2016, 2017, 2018, 2020, 2025 and 2030. New demands in the corresponding years were
 10 inserted into the models, according to the forecast provided by Parc de l' Alba management. In
 11 case of the years for which simulations were not performed (2019, 2021-2024 and 2026-2029), it
 12 was assumed that the energy performance of the DHC plant(s) remains the same as preceding

1 simulation year (energy performance of 2019 is identical to 2018, that of 2021 till 2024 is
2 identical to 2020, while that of 2026 till 2029 is identical to 2025). Since EnergyPRO does not
3 have the capability to dimension the distribution network , LOGSTOR calculator [26] was used
4 for calculating the thermal losses for each year in the different scenarios. The information
5 provided to LOGSTOR is shown in the appendix.

6 It was found that the existing plant is enough to satisfy the demand of 2016. However, in 2017,
7 cooling and heating demands increased by 1768 MWh_{th,c} and 960 MWh_{th,h}, and thus new energy
8 conversion technologies were implemented henceforth. For the future years, different
9 combinations of RES were used to satisfy the demand, as explained before. The best
10 combinations were decided based on the primary energy consumption, CO₂ emissions and NPV.

11 When comparing different fuel types, primary energy consumption is a preferable metric since it
12 considers the energy required to produce one unit of a consumed particular energy on site;
13 including transportation, storage, distribution, delivery and any losses incurred in the process.
14 CO₂ emissions were chosen for evaluating environmental feasibility since they contribute the
15 most to climate change. The primary energy consumption and CO₂ emissions in each scenario
16 were calculated by simply multiplying the specific emissions factors and Primary Energy Factors
17 (PEFs) respectively of each fuel with the total respective fuel consumption in that scenario. Table
18 4 shows the PEFs and CO₂ emission factors of fuels used in various scenarios taken from
19 [27]. These values were assumed to be unchanged for the future years in all scenarios. It should
20 be mentioned here that the low specific emission factors for biogas and biomass might be
21 considered controversial [28]. Even though combustion of biofuels emits carbon that is part of
22 the biogenic carbon cycle, studies and analysis do not always take into account how long does it
23 take for this carbon to return to the biogenic pool from the atmosphere.

1 *Table 4: Primary Energy Factors and Specific emissions factors for fuels used in Parc de l' Alba*

Fuel	Primary Energy Factor	Specific emission factor (kg CO₂/unit fuel)
Imported Electricity	2.37	0.36 kg/kWh
Natural gas	1.20	2.68 kg/m ³
Biogas	0.50	0.62 kg/m ³
Biomass	0.03	0.06 kg/kg

2
3 For economic feasibility, NPV is chosen as the primary indicator, since it takes into account the
4 time value of money for each scenario, thus showing how profitable was one scenario when
5 compared to another. For this purpose, comprehensive economic calculations are performed by
6 creating Profit and Loss (P & L) sheets for each scenario, starting from the present (2016) , up to
7 the end of the concession period (2047). The procedure followed is outlined in detail by Ross et
8 al. [29]. The NPV, assuming an interest rate of 10 %, keeping lifetime of the investment at 31
9 years (with 2016 as year ‘zero’) is calculated by equation (4). The interest rate of 10 % was
10 chosen based on the author`s experience of conducting economic feasibilities of similar projects.

$$NPV = -C_0 + \sum_{t=1}^T \frac{C_t}{(1+i)^t} \quad (4)$$

11 Here, C₀ is the free cash flow in year zero (in €), T is the lifetime of investment (in years), C_t is
12 the free cash flow in year ‘t’ (in €), ‘i’ is the interest rate (in %) and t represents the time of cash
13 flow (in years). Once the EnergyPRO simulations have provided the revenues and operating
14 expenditures, The Free Cash Flow is obtained by solving several equations in a sequential
15 manner as outlined in [29]. Equation (5) shows how the free cash flow is obtained.

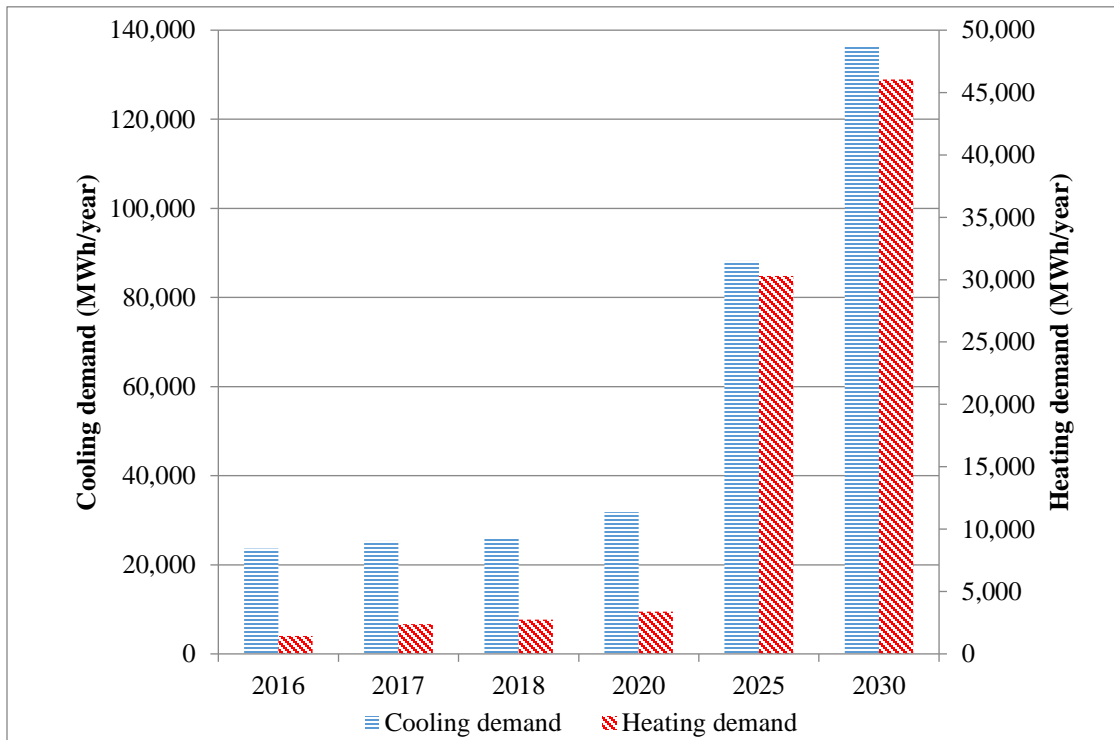
$$Free\ cash\ flow = Operating\ Cash\ Flow - Increase\ in\ Working\ capital - CAPEX \quad (5)$$

16 The Capital Expenditure (CAPEX) refers to the purchases/investments made in each scenario

1 whenever new energy conversion units are introduced, the investment varying as per the
2 technologies being used in the scenario. An inflation of 7.48% was applied to natural gas prices
3 and the variable and fixed prices of heating and cooling ; this value is basically the percentage by
4 which gas prices have increased in Spain in the past 13 years according to IEA [30]. Similarly,
5 electricity prices in Spain have increased by 9.72 % per year on average from 1980 till 2012 and
6 this was the inflation applied to variable price of electricity. An inflation of 2 % is assumed for
7 all other costs and revenues [31].

8 3. Demand modeling

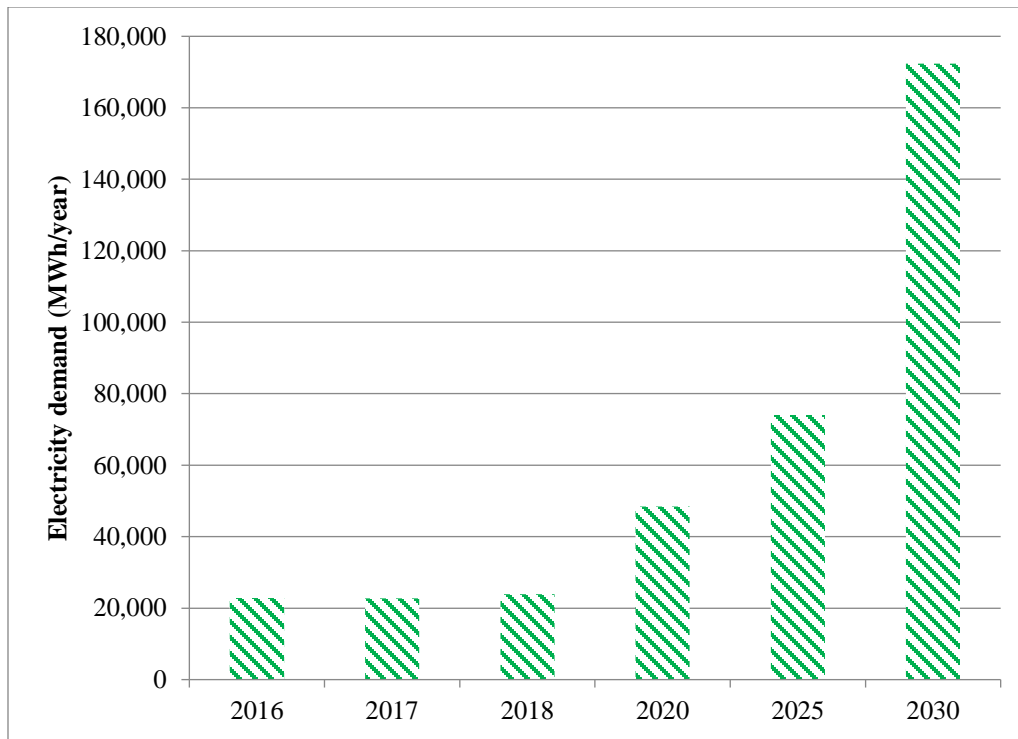
9 The energy demand growth can be seen graphically in Figure 2 and Figure 3 . Future energy
10 demands were provided by the Parc de l' Alba management.



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Figure 2 Cooling and heating demand projection of Parc de l' Alba till 2030

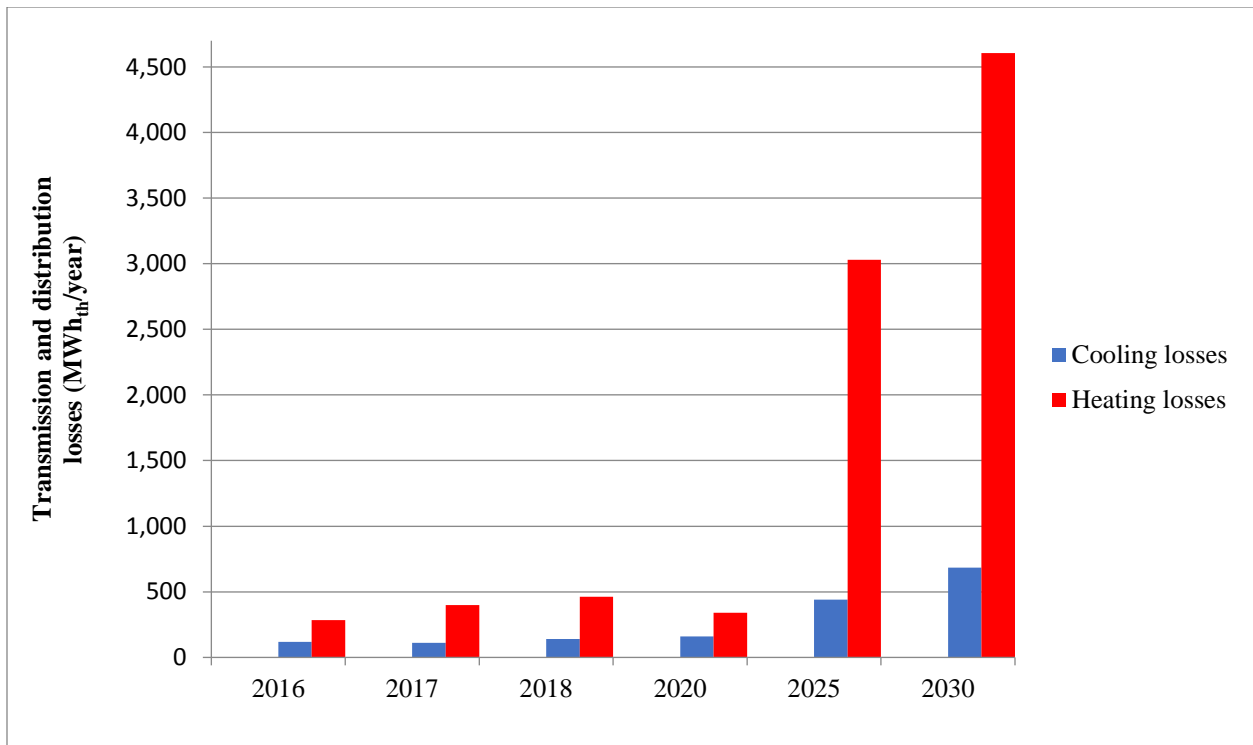


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Figure 3 Electricity demand projection of Parc de l'Alba till 2030

3 The transmission and distribution losses for the cooling and heating lines for Parc de l'Alba are
 4 shown graphically in Figure 4. It is clear that the heating losses are greater than the cooling
 5 losses, indicating that the lines for DH are oversized, in comparison to those for DC.



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Figure 4 Transmission and distribution losses of heating and cooling in Parc de l' Alba till 2030

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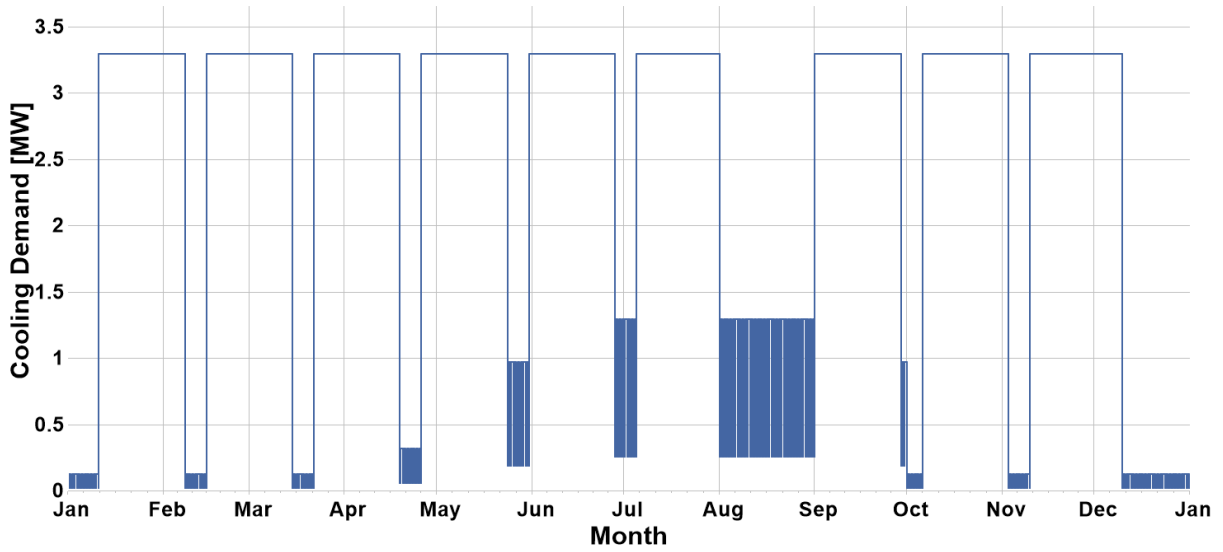
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The different consumers in Parc de l' Alba are the Synchrotron particle accelerator (cooling and electricity), offices (cooling, heating and electricity) and data centers (cooling and electricity). Details of the energy demand growth are shown in the Appendix in tabular form. All energy demands for the various consumers were inserted as a Time Series for each year. The Synchrotron has a periodic operation throughout the year and hence its cooling and electricity demands were modeled according to the data received from Parc de l' Alba. In the first half of the month of January, the full month of August and second half of December, the Synchrotron itself does not require cooling but its offices do, having a peak demand of 1.3 MW_{th,c}. For rest of the year, it operates in cycles: it is in operation for four consecutive weeks (peak demand of 3.3 MW_{th,c}) and out of operation for a week (only its offices require cooling with a peak demand of 1.3 MW_{th,c}). The electricity demand follows a similar behavior to the cooling demand, the main difference being that the electricity demand is expected to increase by five per cent after 2017; this is input on the software by increasing the peak value of electricity consumption when the Synchrotron is in operation. Figure 5 and Figure 6 show the annual cooling and electricity demand profiles respectively of the Synchrotron

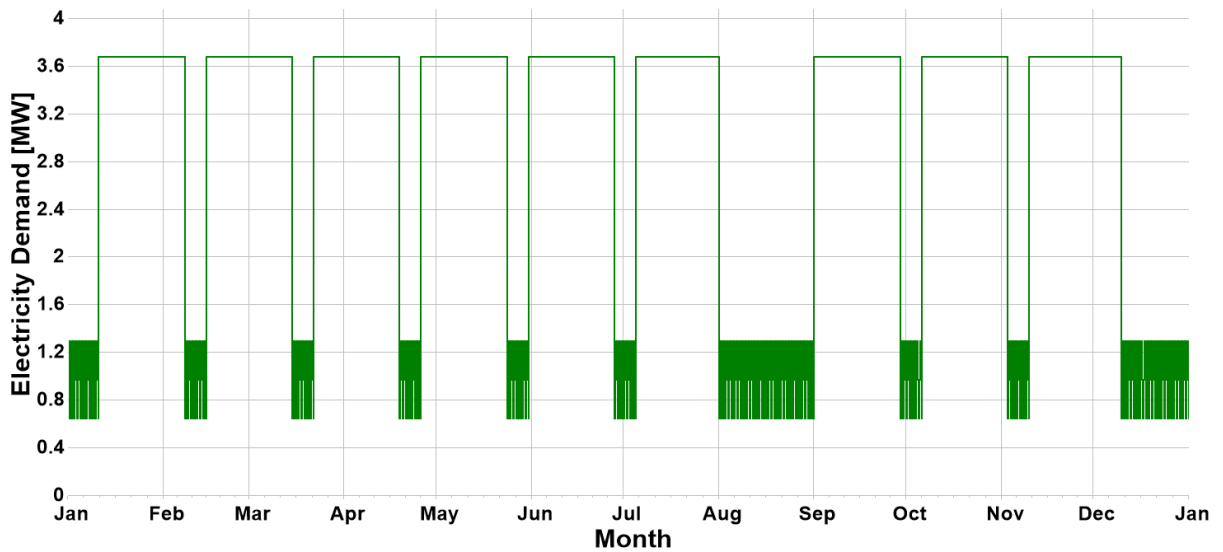
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Figure 5: Annual cooling demand profile of Synchrotron facilities



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Figure 6: Annual electricity demand profile of Synchrotron facilities

6 A notable difference is there between the demand profiles in Figure 5 and Figure 6; this is
7 because electricity demand is not dependent upon ambient conditions and has the same profile
8 throughout the year.

9 For offices, the cooling and heating demands were modeled according to [32] and [33]. The main

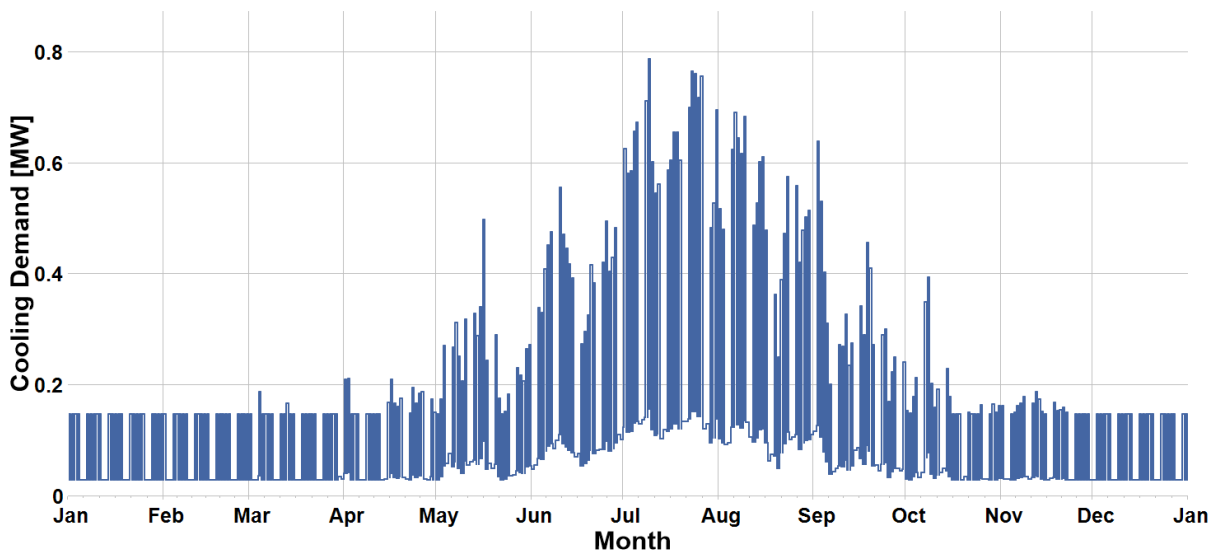
1 assumptions for the cooling demand are:

- 2 • 45-52% of the total demand is linearly dependent on the ambient temperature ; it
3 will start when ambient temperature exceeds 18°C
- 4 • The cooling demand has a fixed profile throughout the year , i.e., 100 % cooling
5 from 6 AM to 8 PM from Mondays to Fridays and 20% at all other times

6 For heating demand, the main assumptions are:

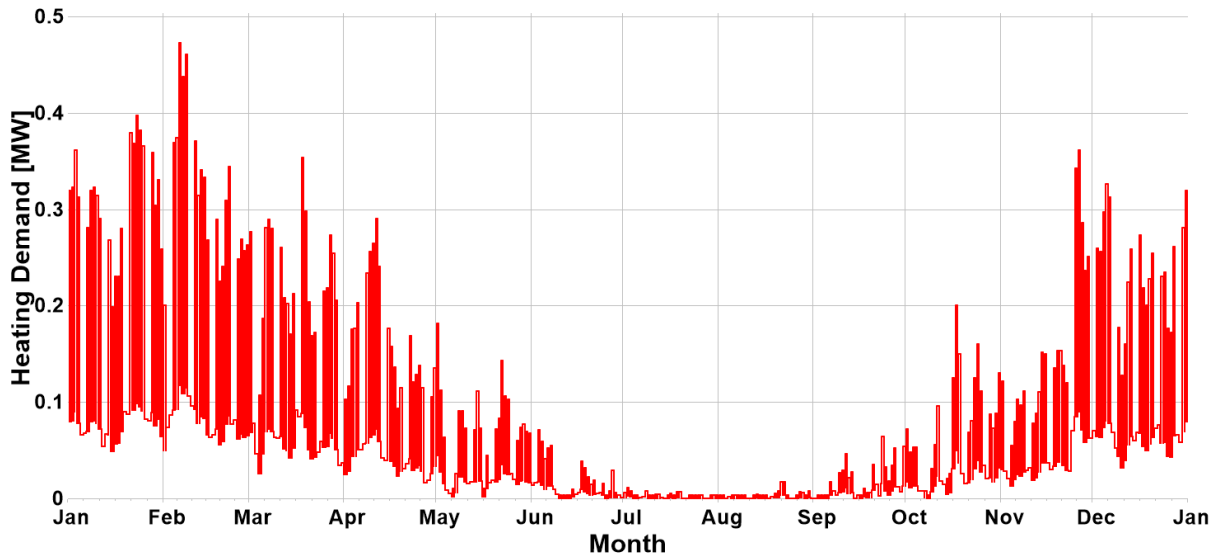
- 7 • 3% of the demand is allocated to Domestic Hot Water , which is provided
8 throughout the year at all times
- 9 • 97% of the demand is linearly dependent on the ambient temperature ; it will be
10 provided when the ambient temperature is below 18°C
- 11 • The heating demand has a fixed profile throughout the year, i.e., 100% heating
12 from 6AM to 8PM from Mondays to Fridays and 25% at all other times

13 Figure 7 and Figure 8 show examples of the annual cooling and heating demand profiles of the
14 offices.



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17 *Figure 7: Annual cooling demand profile of office buildings*



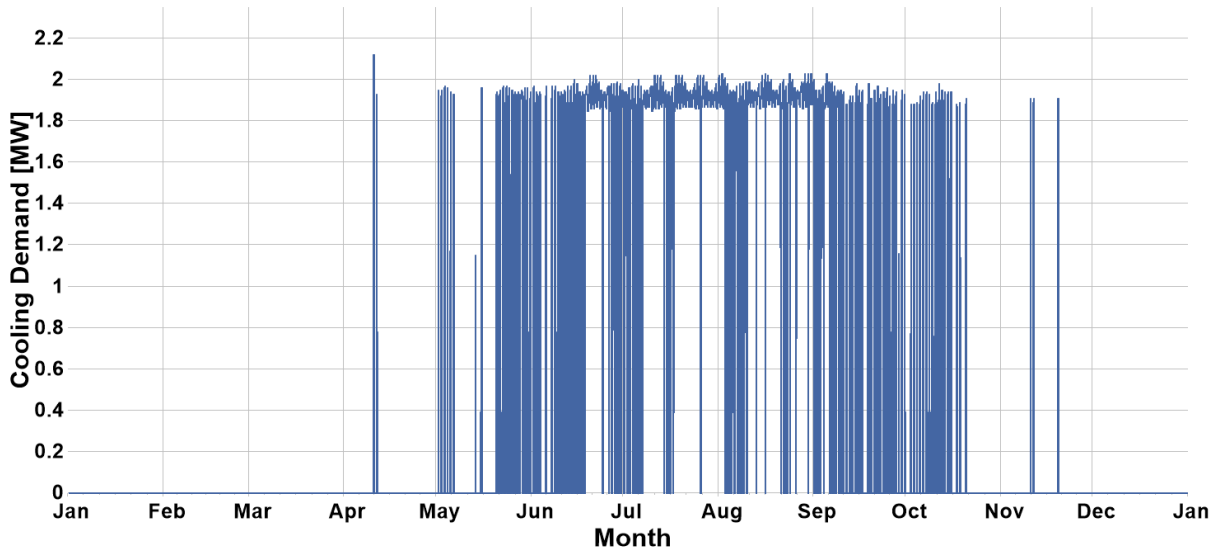
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3 *Figure 8: Annual heating demand profile of office buildings*

4 Finally, cooling and electricity profiles of data centers were characterized on the basis of actual
5 data (operation of IT loads throughout the whole day) simulated in the Renew IT project [34]. To
6 generate the cooling demand profile, TRNSYS software was used. The following inputs were
7 provided to the dynamic simulation models:

- 8
- Type of workload
 - 9 • Contracted cooling power
 - 10 • Set point ambient temperature

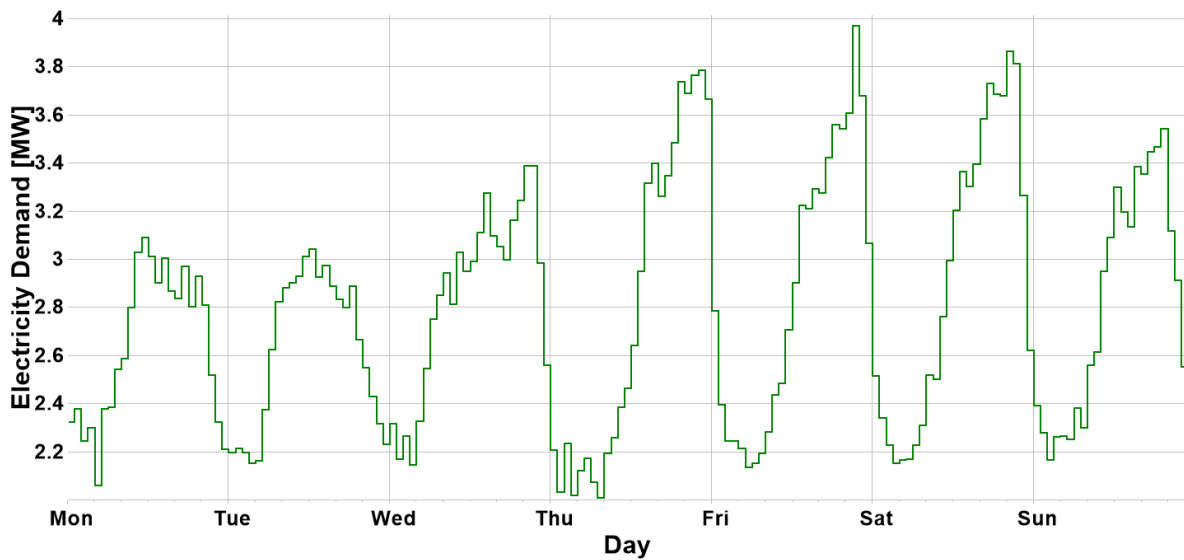
11 It was assumed that all data centers handle only data workload (data analytics, data caching and
12 data serving). Figure 9 and Figure 10 show the annual cooling and weekly electricity demand
13 profiles of data centers generated by simulation. The cooling demand profile is more
14 concentrated in the summer season since the data centers are equipped with free cooling, which
15 is active whenever ambient temperature is below 22°C. The weekly electricity demand remains
16 the same throughout the whole year (as shown in Figure 10).



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Figure 9: Annual cooling demand profile of data centers



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Figure 10: Weekly electricity demand profile of data centers

5 **4. Supply modeling**

6 Three distinct scenarios were considered for supplying the energy demand of Parc de l' Alba. The
 7 choice for the combinations of energy conversion units in the different scenarios were mainly
 8 dictated by the experience of the authors with previous similar DHC networks in Spain. The

1 three scenarios are:

- 2 • Scenario 1: Solar cooling and Biomass cooling (SCBC)
- 3 • Scenario 2: Ground Source Heat Pumps (GSHPs)
- 4 • Scenario 3: Business As Usual (BAU)

5 Common to all scenarios is the incorporation of a biogas boiler in the ST-4 plant in 2017, which
6 is taken out of service in 2020 (due to unavailability of fuel i.e., landfill gas, from 2020 onwards).
7 The biogas boiler has a capacity of $1.5\text{MW}_{\text{th,h}}$, with a thermal efficiency of 90 %.

8 From 2020 to 2030 (both inclusive), the demand is large enough to implement different energy
9 conversion technologies in the new ST-5 plant. The peak power demands dictated the sizing of
10 energy conversion units in these years. In all simulations, the operation strategy provided to the
11 software was such that energy conversion units that produced energy most economically would
12 run first and so on. For sizing the cold-water storage tanks, data was obtained from the Forum
13 and 22@ DHC networks in Barcelona in Spain [25]. A cooling demand to volume ratio of 5
14 $\text{MWh}_{\text{th,c}}/\text{year}/\text{m}^3$ was defined to relate the cooling demand with cold storage size and was hence
15 used to size the cold-water storage tanks in 2020, 2025 and 2030. For hot storage, it was assumed
16 that 20 % of the heating demand would need to be stored, considering a storage time of 12 hours.
17 Table 5 shows details of hot and cold storages added in 2020, 2025 and 2030 in the simulations.

18 *Table 5: Volume and capacities of cold and hot storage tanks at ST-5 plant (2020-2030)*

	2020	2025	2030
Cold water storage tank	6400 m ³ ; 29.8 MWh _{th,c}	11300 m ³ ; 52.4 MWh _{th,c}	9700 m ³ ; 45.0 MWh _{th,c}
Hot water storage tank	200 m ³ ; 5.8MWh _{th,h}	1620 m ³ ; 47.0 MWh _{th,h}	1000 m ³ ; 29.0 MWh _{th,h}

19

20 Another important point to be mentioned is that the existing ST-4 plant still has space for
21 installation of two new cogeneration engines and an absorption chiller. Thus, in all scenarios, it
22 was decided to install two new engines and an absorption chiller in 2025, having specifications

1 shown in Table 6. The back-up natural gas boiler at the ST-4 is only allowed to run in the winter
 2 months for Scenarios 1 and 2, while it can run throughout the whole year for Scenario 3.

3 *Table 6: Technical specifications of equipment installed at ST-4 plant in 2025 (all scenarios)*

Energy Conversion unit	Specifications	Quantity
Cogeneration engines	3350 kW_e ; $3280 \text{ kW}_{th,h}$; $\eta_{el} = 44.9 \%$; $\eta_{th} = 41.1 \%$	2
Double effect absorption chiller	$5000 \text{ kW}_{th,c}$; COP = 1.3	1

4

5 **4.1. Scenario 1: Solar Cooling and Biomass Cooling (SCBC)**

6 For this scenario, it is assumed that the rooftop of the ST-5 plant was used to install parabolic
 7 trough collectors (PTCs). The hot water from these PTCs would be supplied to a double effect
 8 absorption chiller to produce cooling. For the remainder (major) cooling demand and heating
 9 demand, biomass boilers are installed and connected to double effect absorption chillers. Details
 10 of all energy conversion units installed in SCBC scenario are shown in Table 7.

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1 *Table 7: Technical specifications of equipment installed in SCBC scenario (2020-2030)*

Energy Conversion unit	Specifications	Quantity installed each year			Total installed capacity by 2030
		2020	2025	2030	
Parabolic trough collectors	1088 m ² ; 298 kW _{th} (417 kW _{th,c})	1	-	-	298 kW _{th} (417 kW _{th,c})
Absorption chiller (connected to solar collectors)	450 kW _{th,c} ; COP = 1.4	1	-	-	450 kW _{th,c}
Biomass boiler	2,500 kW _{th,h} ; $\eta_{th} = 90\%$ 5,000 kW _{th,h} ; $\eta_{th} = 90\%$	1	1	-	55,000 kW _{th,h}
Absorption chiller(s) (connected to biomass boiler)	3,300 kW _{th,c} ; COP = 1.4 5,000 kW _{th,c} ; COP = 1.4	1	-	1	51,600 kW _{th,c}

2

3 **4.2. Scenario 2: Ground Source Heat Pumps (GSHPs)**

4 In the GSHPs scenario, the cooling and heating demands are supplied by ground source heat
5 pumps (GSHPs) that produce cooling and heating at the same time. The GSHPs produce 3.2
6 units of cooling and 4.2 units of heating, for every unit of electricity input to the compressor
7 (these specifications are assumed to be the same as those of the units used in hospital de Mollet
8 project in Spain [35]). As a consequence of higher heat output in comparison with cooling output
9 for the same input, the GSHPs in this scenario are “oversized” in terms of heating capacity (since
10 cooling demand is always higher than heating demand in Parc de l’ Alba and it is assumed that
11 there is no drift in soil temperature occurring due to imbalance between the two types of thermal
12 loads). Since GSHPs require very large areas of land on-site for digging boreholes, no new
13 GSHPs are installed in 2030 and the new cooling and heating demands for that year are supplied
14 by biomass boilers connected to double effect absorption chillers. Details of all energy
15 conversion units installed in GSHPs scenario are shown in Table 8.

1 *Table 8: Technical specifications of equipment installed in GSHPs scenario (2020-2030)*

Energy Conversion unit	Specifications	Quantity installed each year			Total installed capacity by 2030
		2020	2025	2030	
Solar PV system	1800 m ² ; 275 kW _e ; $\eta_{el} = 1$ 5.4%, Type: Monocrystalline	1	-	-	275 kW _e
Ground Source Heat Pumps	Input: 1214 kW _e ; Output: 3900 kW _{th,c} , 5113 kW _{th,h} Input: 9,368 kW _e ; Output: 30,100 kW _{th,c} , 39461 kW _{th,h}	1	-	-	34,00 kW _{th,c} , 5,113 kW _{th,h}
Biomass boilers	5000 kW _{th,h} ; $\eta_{th} = 90\%$	-	-	5	25,000 kW _{th,h}
Absorption Chillers	5000 kW _{th,c} ; COP = 1.4	-	-	5	25,000 kW _{th,h}

2

3 **4.3. Scenario 3: Business as Usual (BAU)**

4 In the BAU scenario, cogeneration engines are installed to sell electricity directly to the data
5 centers from 2020 onwards (as opposed to buying it from the grid and then selling to the
6 consumers as modelled in the previous scenarios). Exhaust heat from the engines would be used
7 to satisfy the heating demand and would also be supplied to absorption chillers for producing
8 cooling. For backup, natural gas boilers and a compression chiller are installed as well. Details of
9 all energy conversion units installed in BAU scenario are shown in Table 14.

1 *Table 9: Technical specifications of equipment installed in BAU scenario (2020-2030)*

Energy Conversion unit	Specifications	Quantity installed each year			Total installed capacity by 2030
		2020	2025	2030	
		Cogeneration engines	3,350 kW _e ; 3,280 kW _{th,h} ; $\eta_{el} = 44.9 \%$; $\eta_{th} = 41.1 \%$	3	
Double effect absorption chillers	5,000 kW _{th,c} ; COP = 1.3	1	1	2	20,000 kW _{th,c}
Single effect absorption chillers	3,000 kW _{th,c} ; COP = 0.75	-	2	1	9,000 kW _{th,c}
Natural gas boilers	5,000 kW _{th,h} ; $\eta_{th} = 60\%$	1 (back up)	2	-	15,000 kW _{th,h}
Compression chiller	15,000 kW _{th,c} ; COP = 4	-	1	-	15,000 kW _{th,c}

2

3 5. Economic modeling

4 Fuel prices provided to the EnergyPRO models from 2016 to 2030 are shown in the Appendix.
5 Prices of natural gas and electricity have been inflated according to the percentages explained in
6 section 2 while landfill gas and biomass are inflated at 2% (inflation rate of the Euro region). The
7 price and maintenance costs of the biogas boiler (having a capacity of 1.5 MW_{th,h}), installed in
8 2017, are € 147,500 and € 6,250 respectively. Investment costs of various equipment in the
9 different scenarios were available for present day and are inflated at 2% every year, so as to get
10 the expected cost of the actual year in which they were integrated in the models.

11 Table 10: Investment costs of cold and hot storage tanks at ST-5 plant shows costs of the cold [25]
12 and hot storage tanks [36] installed at ST-5 from 2020 to 2025. The specifications of these
13 storage tanks have been shown previously shown in Table 5.

1 *Table 10: Investment costs of cold and hot storage tanks at ST-5 plant*

	2020	2025	2030
Cold water storage tank	€ 1,400,000	€ 2,700,000	€ 2,600,000
Hot water storage tank	€ 97,400	€ 290,000	€ 237,500

2

3 Cost of the new equipment installed at ST-4 plant in 2025 is shown in Table 11. Cost of engines
4 and double effect absorption chillers have been provided by the management at Parc de l' Alba

5 *Table 11: Investment cost of engines and chiller at ST-4 plant in 2025*

Energy Conversion unit	Specifications	Total investment
Cogeneration engines	3350 kW _e ; 3280 kW _{th,h} ; $\eta_{el} = 44.9 \%$	€ 3,700,000
Double effect absorption chiller	5000 kW _{th,c} ; COP = 1.3	€ 1,100,000

6

7 For the ST-5 plant, the CAPEX of the SCBC, GSHPs and BAU scenarios are shown in Table 12,
8 Table 13 and Table 14 respectively.

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Table 12: Investment costs of energy conversion units installed in SCBC scenario

Energy Conversion unit	Total investment		
	2020	2025	2030
Parabolic trough collectors	€ 353,000	-	-
Absorption chiller (connected to solar collectors)	€ 277,000	-	-
Auxiliary equipment for solar thermal cooling system	€ 631,000	-	-
	€ 1,172,000	€1,293,000	
Biomass boiler	-	€12,940,000	€ 14,280,000
Absorption Chillers	€ 659,000	-	€ 803,000
(connected to biomass boilers)	-	€ 6,000,000	€ 3,650,000

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Table 13: Investment costs of energy conversion units installed in GSHPs scenario

Energy Conversion unit	Total Investment		
	2020	2025	2030
Solar PV system	€390,000	-	-
	€8,762,000	-	-
Ground Source Heat Pumps	-	€74,650,000	-
Biomass boilers	-	-	€14,280,000
Absorption Chillers	-	-	€6,070,000

1

Table 14: Investment costs of energy conversion units installed in BAU scenario

Energy Conversion unit	Total Investment		
	2020	2025	2030
Cogeneration engines	€4,931,000	€1,815,000	€10,020,000
Double effect absorption chillers	€998,000	-	€ 1,216,000
Single effect absorption chillers	-	€1,322,000	€1,460,000
Natural gas boilers	€123,000	€272,000	-
Compression chiller	-	€1,484,000	-

2

3 Cost of the solar collectors was provided by a manufacturer based in Sweden [37], while the
4 price of the absorption chiller was taken from [38]. For the GSHPs scenario, investment and
5 maintenance costs were taken from a study carried out in France [39], including cost of drilling
6 in the ground and construction of boreholes. Costs of the solar PV system were taken from IEA
7 [40]. In the BAU scenario, investment costs of engines and natural gas boilers were provided by
8 Parc de l' Alba management, while cost of the compression chiller was calculated from a guide
9 published by the Catalan Energy Institute [41].

10 6. Results and Discussion

11 This section presents key results of the EnergyPRO simulations and economic analysis. Results
12 are shown in a chronological order, i.e., first of the years 2016 till 2018 and then of 2020 till
13 2030 for the different scenarios.

14 6.1. Energy Performance

15 From 2016 till 2018, all three scenarios have identical simulation models. Energy balance of
16 2016-2018 is shown in Table 15. Energy demand may be seen in Figure 2 and Figure 3 in section
17 3. It must be stressed here that energy consumption refers to the actual amount of energy

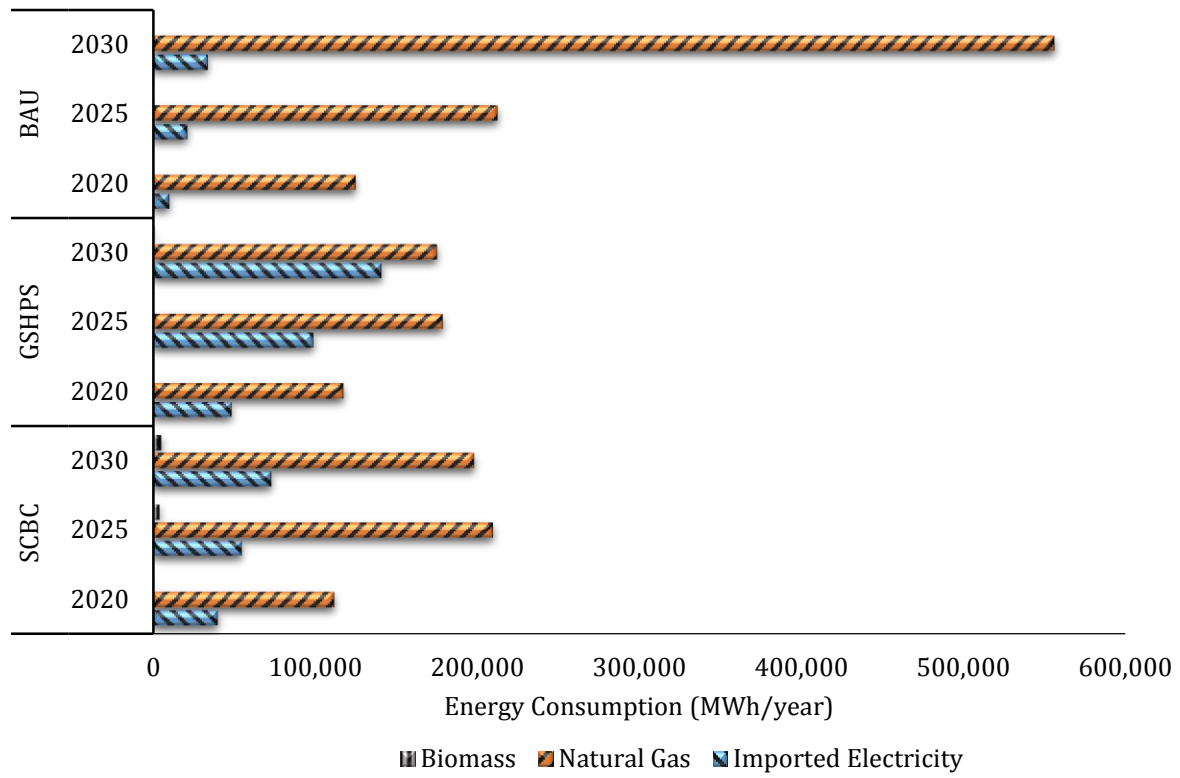
1 consumed while primary energy consumed is the total energy required to produce one unit of a
 2 consumed particular energy on site; including transportation, storage, distribution, delivery and
 3 any losses incurred in the process.

4 *Table 15: Energy Balance at Parc de l' Alba from 2016 until 2018*

	2016	2017	2018
Energy Consumption(MWh/year)			
Imported Electricity	16,800	17,000	17,000
Natural gas	92,900	94,900	92,000
Biogas	-	9,200	9,100
Primary Energy Consumption(MWh/year)			
Imported electricity	39,900	40,200	41,000
Natural gas	111,000	113,000	110,000
Biogas	-	4,600	4,600
Total	151,000	158,000	156,000

5
 6 Even though the energy consumption has increased from 2017 to 2018, the primary energy consu
 7 mption has decreased since there is a reduction in the use of natural gas by the backup boiler. Thi
 8 s backup boiler not allowed to operate during the summer and priority was given to the more effi
 9 cient biogas boiler, burning a cheaper fuel.

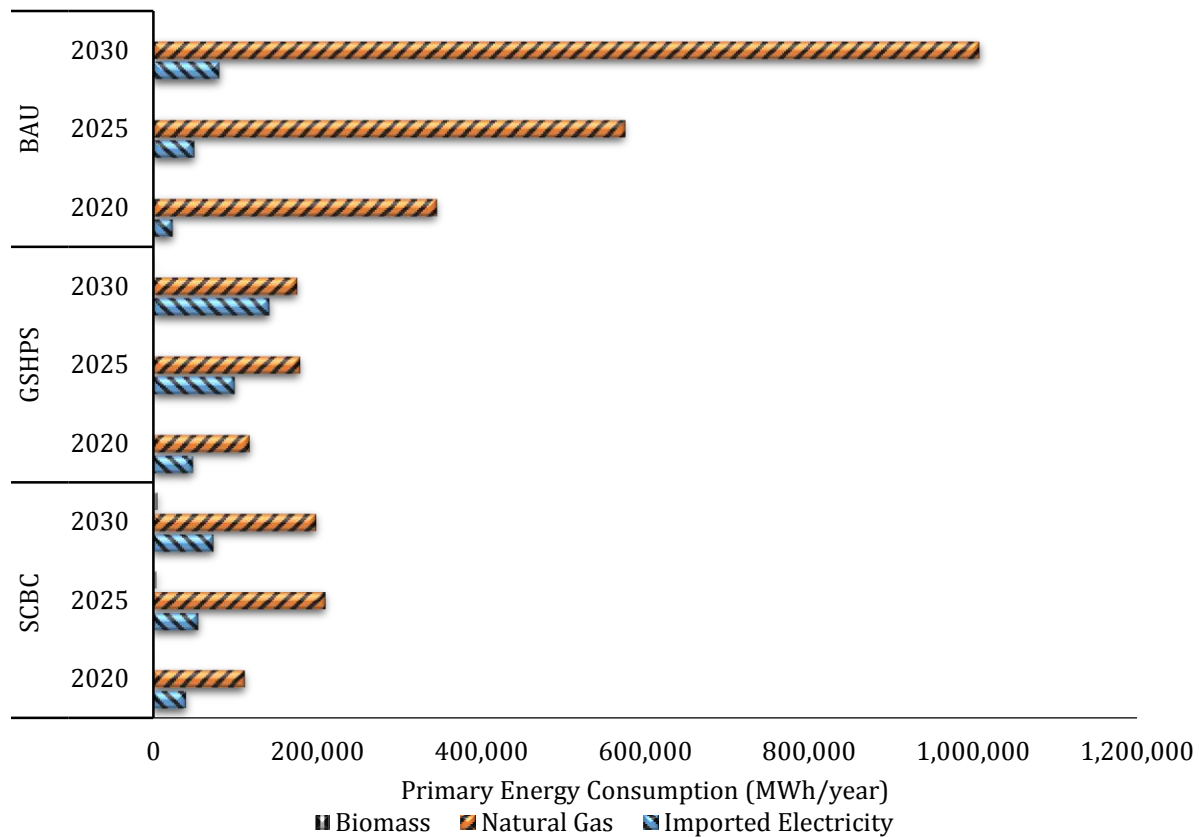
10 2020 is the year when increase in energy demand is large enough to justify development of the
 11 new ST-5 plant, where different energy conversion technologies are implemented in accordance
 12 with the different scenarios (SCBC, GSHPs and BAU). Energy balances from 2020 till 2030 are
 13 shown in Figure 11 and Figure 12.



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Figure 11: Energy Balance for different scenario of the basis of energy consumption (MWh/year)



1

2 *Figure 12: Energy Balance for different scenarios on the basis of primary energy consumption*
 3 *(MWh/year)*

4 By comparison between Figure 11 and Figure 12, it can be seen that the GSHPs scenario has a
 5 larger energy and primary energy consumption as compared the SCBC scenario. This is because
 6 of the fact that the former relies more on electricity to run the GSHP units while the latter uses
 7 biomass as the main fuel, which has a lower PEF as shown previously in Table 4 in section 2.

8 The BAU scenario has a drastically larger consumption of energy, which is clearly due to
 9 burning of much more natural gas. The new ST-5 plant in this case relies only on the new
 10 cogeneration engines for energy production. Moreover, the old engines in the ST-4 plant are
 11 running more hours in comparison with SCBC and GSHPs scenario, since there are no RES
 12 based energy conversion units in BAU scenario. Moreover, as explained previously, in the BAU
 13 scenario the backup boiler at the ST-4 plant is allowed to operate throughout the year and not

1 only in winter months as in the SCBC and GSHPs scenario, where priority is to run the RES
2 based units.

3 **6.2. Environmental analysis**

4 Table 16 shows the CO₂ emissions from 2016 until 2018, while emissions from 2020 until 2030
5 are shown in Table 17.

6 *Table 16: CO₂ emissions in Parc de l'Alba from 2016 until 2018*

	2016	2017	2018
CO₂ emissions (tons/year)	29,400	30,900	30,400

7

8 *Table 17: CO₂ emissions in Parc de l'Alba from 2020 till 2030 (all scenarios)*

	2020	2025	2030
SCBC scenario (tons/year)	38,600	71,800	107,800
GSHPs scenario (tons/year)	40,800	70,300	111,170
BAU scenario (tons/year)	76,300	128,500	224,500

9

10 From Table 17, BAU scenario shows more than twice the amount of CO₂ emissions in 2030,
11 mainly because it is burning more natural gas in comparison to the RES based scenarios, due to
12 the new cogeneration engines and new boilers at ST-5 plant and longer running hours of the old
13 engines and old boiler at the ST-4 plant.

14 **6.3. Economic results**

15 The major economic indicators obtained from the EnergyPRO simulations and the P & L
16 analysis are mainly the revenues, operating costs, EBITDA (Earnings before Interests, Taxes,
17 Depreciation and Amortization) and NPV. Figure 13 show all these indicators (except NPV) for
18 2016-2018 and 2020-2030 respectively.

1

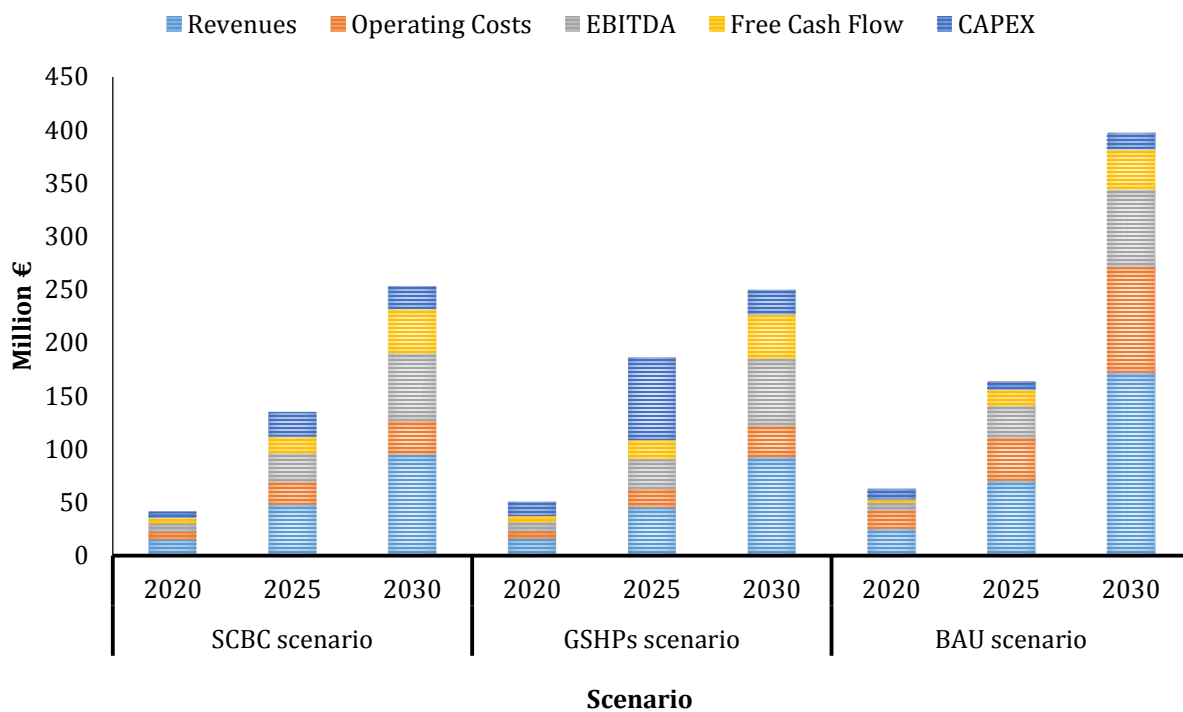
Table 18: Economic indicators of Parc de l' Alba from 2016 till 2018

	2016	2017	2018
Revenues	€ 7,651,000	€ 8,820,000	€ 10,155,000
Operating Costs	€ 5,080,000	€ 5,604,000	€ 5,887,000
EBITDA	€ 2,571,000	€ 3,216,000	€ 4,268,000
Free Cash Flow	€ 629,000	€ 1,962,000	€ 2,722,000

2

3 The NPV is calculated until the end period for each scenario and is thus shown graphically in

4 Figure 14: Net Present Value (NPV) of each scenario for each scenario.



5

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Figure 13: Economic indicators of the different scenarios

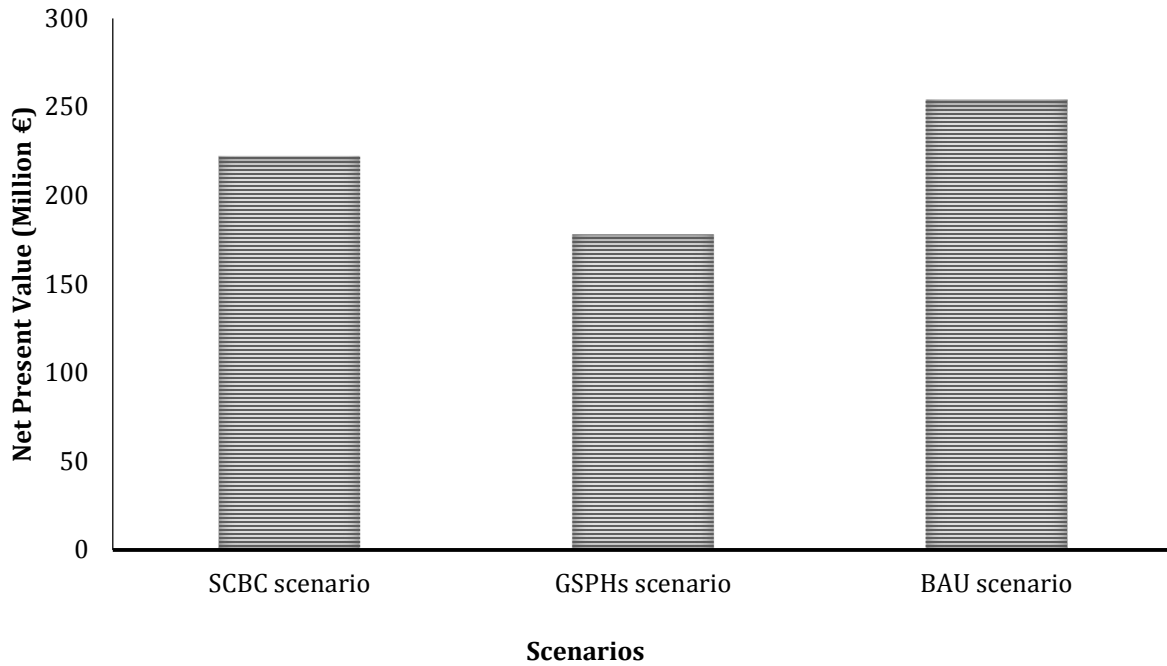


Figure 14: Net Present Value (NPV) of each scenario

By comparing the two RES based scenarios from Figure 13 and Figure 14, it can be seen clearly that the SCBC scenario is more profitable in comparison to GSHPs scenario, due to its lower CAPEX and higher NPV. Apparently, GSHPs have an average efficiency which is four times that of boilers burning biomass and providing heat to absorption chillers in the SCBC scenario. However, the CAPEX in the GSHPs scenario is much higher than SCBC scenario and moreover, cost of imported electricity is higher than biomass and it increases at a faster rate (than that of biomass), as explained in section 2.

When doing the comparison as a whole, it is indeed the BAU scenario which is most profitable. Not only does it have the lowest CAPEX in comparison to the RES based scenarios, but it has an additional revenue stream generated by exporting excess electricity produced to the grid.

Conclusion

The paper presents a technical, economic and environmental evaluation of implementing different RES based systems in the expansion of a fossil fuels based DHC plant in a Technology Park in the region of Catalonia in Spain. The expansion of the existing plant was dictated by the forecast of increased demand of cooling, heating and electricity in the Technology Park. The aim

1 of the study was to evaluate which combination of RES based systems would be best suited for
2 integrating into the expansion of the DHC plant.

3 In context of the given assumptions and input parameters, the results of the simulations and
4 economic evaluation showed that when deciding between RES based technologies, SCBC
5 scenario is more feasible option when compared to the GSHPs scenario, due to its lower primary
6 energy consumption (624,380 MWh/year in 2030 vs. 665,367 MWh/year), lower CO₂ emissions
7 (107,753 tons/year in 2030 vs. 111,166 tons/year) and higher NPV (222 million € vs. 178 million
8 €) with lower CAPEX.

9 Considering the SCBC scenario, solar thermal cooling has great prospects for primary energy
10 savings and reduction of CO₂ emissions but in this study , the limited area of ST-5 plant`s roof
11 was the reason that solar thermal cooling could not contribute a lot in the large cooling demand
12 of Parc de l'Alba. Only 2% of the cooling demand in 2020 came from the solar thermal system,
13 dropping to a mere 1% in 2025 and 2030. This was not surprising because the maximum output
14 of the solar thermal cooling system installed was 417 kW_{th,c}, whereas the total cooling capacity
15 of ST-5 plant was 3717 kW_{th,c}, 39,717 kW_{th,c} and 58,017 kW_{th,c} in 2020, 2025 and 2030
16 respectively. In the GSHPs scenario, the units are producing more heating than cooling, as
17 explained in section 4.2. For Parc de l'Alba, this excess heat is useless and hence rejected to the
18 soil. Basically, implementation of GSHPs for simultaneous production of cooling and heating is
19 not feasible for office buildings and data centers located in a Mediterranean climate (Catalonia),
20 where cooling is the main energy demand and not heating. Additionally, digging of boreholes for
21 installation of GSHPs requires large land areas, an additional investment that has to be taken into
22 account.

23 In contrast to solar thermal cooling and GSHPs, biomass boilers burning woodchips connected to
24 double effect absorption chillers are apparently the most favorable solution in this study. Not
25 only is biomass cheaper than importing electricity, but there are large savings in primary energy
26 consumption as well (wood chips and electricity have PEFs of 0.034 and 2.368 respectively as
27 shown in table 7). Moreover, for the same cooling capacities, biomass boilers and absorption
28 chillers need less land area for installation as compared to GSHPs and solar thermal cooling
29 systems.

1 Thus, in the frame of this study, based on the given assumptions and inputs, biomass boilers
2 connected to absorption chillers with assistance from solar thermal cooling are the most feasible
3 renewable energy system technology, for the large district heating and cooling network of Parc
4 de l'Alba in Spain.

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1 **References**

- 2 [1] European Commission, “2030 climate & energy framework _ Climate Action,”
3 2017. .
- 4 [2] European Comission, “Buildings - European Commission,” *Energy*, 2014. .
- 5 [3] European Commission, “EUR-Lex - 32012L0027 - EN - EUR-Lex,” 2015. .
- 6 [4] H. Dagdougui, R. Minciardi, A. Ouammi, M. Robba, and R. Sacile, “Modeling and
7 optimization of a hybrid system for the energy supply of a ‘green’ building,”
8 *Energy Convers. Manag.*, vol. 64, pp. 351–363, 2012.
- 9 [5] H. Wang, W. Yin, E. Abdollahi, R. Lahdelma, and W. Jiao, “Modelling and
10 optimization of CHP based district heating system with renewable energy
11 production and energy storage,” *Appl. Energy*, vol. 159, pp. 401–421, 2015.
- 12 [6] P. A. Østergaard, “Comparing electricity, heat and biogas storages’ impacts on
13 renewable energy integration,” *Energy*, vol. 37, no. 1, pp. 255–262, 2012.
- 14 [7] S. Nielsen and B. Möller, “Excess heat production of future net zero energy
15 buildings within district heating areas in Denmark,” *Energy*, vol. 48, no. 1, pp. 23–
16 31, 2012.
- 17 [8] G. Streckiene, V. Martinaitis, A. N. Andersen, and J. Katz, “Feasibility of CHP-
18 plants with thermal stores in the German spot market,” vol. 86, pp. 2308–2316,
19 2009.
- 20 [9] H. Lund, G. Šiupšinskas, and V. Martinaitis, “Implementation strategy for small
21 CHP-plants in a competitive market: The case of Lithuania,” *Appl. Energy*, vol. 82,
22 no. 3, pp. 214–227, 2005.
- 23 [10] O. Arslan and R. Kose, “Exergoeconomic optimization of integrated geothermal
24 system in Simav, Kutahya,” *Energy Convers. Manag.*, vol. 51, no. 4, pp. 663–676,
25 2010.

- 1 [11] M. De Carli, A. Galgaro, M. Pasqualetto, and A. Zarrella, “Energetic and economic
2 aspects of a heating and cooling district in a mild climate based on closed loop
3 ground source heat pump,” *Appl. Therm. Eng.*, vol. 71, no. 2, pp. 895–904, 2014.
- 4 [12] A. Hakkaki-Fard, P. Eslami-Nejad, Z. Aidoun, and M. Ouzzane, “A techno-
5 economic comparison of a direct expansion ground-source and an air-source heat
6 pump system in Canadian cold climates,” *Energy*, vol. 87, pp. 49–59, 2015.
- 7 [13] G. Angrisani, G. Diglio, M. Sasso, F. Calise, and M. Dentice d’Accadia, “Design of
8 a novel geothermal heating and cooling system: Energy and economic analysis,”
9 *Energy Convers. Manag.*, vol. 108, pp. 144–159, 2016.
- 10 [14] M. Tan and A. Keçebas, “Thermodynamic and economic evaluations of a
11 geothermal district heating system using advanced exergy-based methods,” vol. 77,
12 pp. 504–513, 2014.
- 13 [15] A. Keçebaş and A. Hepbasli, “Conventional and advanced exergoeconomic
14 analyses of geothermal district heating systems,” *Energy Build.*, vol. 69, pp. 434–
15 441, 2014.
- 16 [16] A. Baghernejad, M. Yaghoubi, and K. Jafarpur, “Exergoeconomic optimization and
17 environmental analysis of a novel solar-trigeneration system for heating, cooling
18 and power production purpose,” *Sol. Energy*, vol. 134, pp. 165–179, 2016.
- 19 [17] D. Buoro, P. Pinamonti, and M. Reini, “Optimization of a Distributed Cogeneration
20 System with solar district heating,” *Appl. Energy*, vol. 124, pp. 298–308, 2014.
- 21 [18] M. F. Torchio, “Comparison of district heating CHP and distributed generation
22 CHP with energy, environmental and economic criteria for Northern Italy,” *Energy
23 Convers. Manag.*, vol. 92, pp. 114–128, 2015.
- 24 [19] V. M. Soltero, “Evaluation of the potential for District Heating Cogeneration in
25 Spain as a tool for economy decarbonization,” pp. 1–21, 2016.

- 1 [20] A. Kazagic, A. Merzic, E. Redzic, and D. Tresnjo, “Optimization of modular
2 district heating solution based on CHP and RES - Demonstration case of the
3 Municipality of Visoko,” *Energy*, vol. 181, pp. 56–65, 2019.
- 4 [21] M. Rämä and M. Wahlroos, “Introduction of new decentralised renewable heat
5 supply in an existing district heating system,” *Energy*, vol. 154, pp. 68–79, 2018.
- 6 [22] M. E. Martínez and C. Martín-gómez, “Reasons Why District Energy Systems
7 Were Not Extended in Spain,” in *39th World Congress on Housing Science
8 Changing Needs, Adaptive Buildings, Smart Cities. Politecnico di Milano, Italy,*
9 2013.
- 10 [23] EMD International A/S, “energyPRO.” .
- 11 [24] “• Spain: Electricity prices for households 2010-2018 | Statistic.” [Online].
12 Available: [https://www.statista.com/statistics/418085/electricity-prices-for-](https://www.statista.com/statistics/418085/electricity-prices-for-households-in-spain/)
13 [households-in-spain/](https://www.statista.com/statistics/418085/electricity-prices-for-households-in-spain/). [Accessed: 08-Jun-2019].
- 14 [25] C. Dapena, “Consorci Urbanístic del Centre Direccional de Cerdanyola del
15 Vallès ,Parc de l’Alba (personal communication).” 2016.
- 16 [26] LOGSTOR, “Calculator for insulation on LOGSTOR pipes.” [Online]. Available:
17 <https://www.logstor.com/service-support/tools/logstor-calculator>. [Accessed: 09-
18 Jun-2019].
- 19 [27] E. y T. Ministerio de Industria, “Factores de emisión de CO2 y coeficientes de paso
20 a energía primaria de diferentes fuentes de energía final consumidas en el sector de
21 edificios en España (in Spanish),” 2014.
- 22 [28] J. S. Gunn, D. J. Ganz, and W. S. Keeton, “Biogenic vs. geologic carbon emissions
23 and forest biomass energy production,” *GCB Bioenergy*, vol. 4, no. 3, pp. 239–242,
24 2012.
- 25 [29] S. Ross, R. Westerfield, and B. Jordan, *Fundamentals of Corporate Finance*, 11th

- 1 ed. New York: McGraw-Hill, 2014.
- 2 [30] International Energy Agency (IEA), *Energy Policies of IEA Countries - Spain 2015*
3 *Review*. 2015.
- 4 [31] European Central Bank, “Monetary Policy,” 2010. .
- 5 [32] Ajuntament de Barcelona, “Pla de Millora Energética de Barcelona,” Barcelona,
6 2002.
- 7 [33] R. Ulseth and L. Pedersen, “Method for load modelling of heat and electricity
8 demand,” in *10th International Symposium on District Heating and Cooling*, 2006,
9 no. September.
- 10 [34] European Commission, “Energy requirements for IT equipment - Renewit :
11 Renewit.” .
- 12 [35] FuturENERGY, “Hospital de Mollet, energy efficiency and sustainability -
13 FuturEnergy,” 2016. .
- 14 [36] T. Schmidt and O. Miedaner, “Solar district heating guidelines - Storage,” *Solar*
15 *district heating guidelines*, 2012. .
- 16 [37] A. Rammsy, “Absolicon Solar Collectors AB (personal communication, 2016).”
17 Stockholm, 2016.
- 18 [38] A. Shirazi, R. A. Taylor, S. D. White, and G. L. Morrison, “A systematic parametric
19 study and feasibility assessment absorption chillers for heating and cooling
20 applications,” *ENERGY Convers. Manag.*, vol. 114, pp. 258–277, 2016.
- 21 [39] C. Boissavy, “Cost and Return on Investment for Geothermal Heat Pump Systems
22 in France,” 2015, no. April, pp. 19–25.
- 23 [40] International Energy Agency (IEA), “Technology Roadmap - Solar Photovoltaic
24 Energy,” 2014.

1 [41] Institut Català d’Energia (ICAEN), “Guia de desenvolupament de projectes de
2 xarxes de Districtes de Calor i Frio,” 2010.

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1 **Appendix**

2 For the ST-4 plant at Parc de l' Alba, revenues from heating, cooling and electricity sales
3 comprise of capacity payments and a variable price. Additionally, heating and cooling sale
4 revenues comprise of a connection payment. The connection costs are paid just once whenever a
5 new consumer in Parc de l' Alba signs an agreement to buy heating and cooling from the plant.
6 The capacity payment refers to payment made every year by the consumer in accordance with
7 the power they have contracted from the plant. Finally, the variable price is payment made by the
8 consumer for each unit of energy purchased. Table 19 and Table 20 show details of all these
9 revenues for the year 2015.

10 *Table 19: Revenues from heating and cooling sales at Parc de l' Alba for base model (2015)*

Payment type	Value for cooling	Value for heating
Connection payment	14,580 €/MW _{th,c} connected	48,200 €/MW _{th,h} connected
Capacity payment	23,000€/MW _{th,c} /year	14,000 €/MW _{th,h} /year
Variable price	34.8 €/MWh _{th,c} sold	34.8 €/MWh _{th,h} sold

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12 *Table 20: Revenues from electricity sales at Parc de l' Alba for base model (2015)*

Payment type	Value for electricity (Synchrotron)	Value for electricity (grid)
Capacity payment	617,130 €/year	529,200 €/year
Variable price	114.2 €/MWh _e sold	113.2 €/MWh _e sold

13

14 All expenses of the plant are shown in Table 21 and Table 22 for 2015. Note that Parc de l' Alba
15 pays only the marginal electricity production cost when it buys from the electric grid and hence
16 the large difference between the revenue it earns per unit energy by selling to the grid, compared
17 to what it pays per unit when it needs to purchase from the grid.

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Table 21: Fuel expenses at Parc de l' Alba in 2015

Fuel	Value
Natural gas	37.8 €/MWh _{th} (0.402 €/m ³)
Electricity imported	40.7 €/MWh _e

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Table 22: Maintenance expenses at Parc de l' Alba in 2015

Maintenance type	Value
Fixed maintenance	245,100 €/year
Variable maintenance	13.3 €/MWh _e from gas engines
Overhaul of engines	6.8 €/hour operation

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6 Fuel prices provided to the EnergyPRO models from 2016 to 2030 are shown in Table 23 [25].

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Table 23: Fuel prices in Parc de l' Alba from 2016 till 2030

Year	2016	2017	2018	2020	2025	2030
Natural gas (€/kWh_{th})	0.040	0.043	0.046	0.050	0.072	0.104
Imported electricity (€/kWh_e)	0.041	0.045	0.049	0.060- 0.068	0.094- 0.129	0.149- 0.212
Landfill gas (€/kWh_{th})	-	0.0082	0.0083	-	-	-
Biomass (€/kWh_{th})	-	-	-	0.033	0.036	0.040

1 The specifications of different categories of energy consumers at Parc de l' Alba, including
 2 current and future ones, are shown in Table 24 , including the year in which they will be
 3 connected to the DHC network.

4 *Table 24: Details of expected energy consumers of Parc de l' Alba till 2030*

Consumer name in software Model	Consumer type	Energy service(s) provided	Year of connection	Number of building(s)	Total floor area (m²)
Synchrotron	Particle Accelerator	Heating, Cooling, Electricity	2010	1	35,000
Plot 1	Offices	Heating, Cooling	2013	1	43,000
Plot 2	Data Center	Cooling	2016	1	51,230
Plot 3	Offices	Heating, Cooling	2017	1	81,230
Plot 4	Offices	Heating, Cooling	2018	1	89,230
Plots 2020-DC	Data Center	Cooling, Electricity	2020	1	93,230
Plots 2020-Off	Offices	Heating, Cooling	2020	2	108,230
Plots 2025-DC	Data Center	Cooling, Electricity	2025	1	112,230
Plots 2025-Off	Offices	Heating, Cooling	2025	51	732,000
Plots 2030-DC	Data Center	Cooling, Electricity	2030	1	736,000
Plots 2030-Off	Offices	Heating, Cooling	2030	31	112,2000

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 6 EnergyPRO does not have the capability to dimension the distribution network because the
 7 return and supply temperatures of the fluids in the network cannot be input to the simulation
 8 models. For this purpose, LOGSTOR calculator, which is an internet-based program, was used

1 for calculating the heating and cooling line losses. The major information used for calculating the
 2 losses is shown in Table 25 . Note that the various sections of the DHC network had varying pipe
 3 diameters.

4 *Table 25: Parameters for calculating transmission and distribution losses for Parc de l' Alba*

Parameter		Value
	Number of summer days	182
	Number of winter days	183
	Summer ambient temperature	19.5°C
	Winter ambient temperature	11.6°C
	Soil cover	1500 mm
	Soil thermal conductivity	1.6 W/m-K
	Pipe material	Steel
	Supply temperature	90°C
District heating	Return temperature	75°C
	Internal pipe diameters (mm)	100/125/150/500
	Supply temperature	5°C
District cooling	Return temperature	12°C
	Internal pipe diameters (mm)	150/200/300/400/500/700/800

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