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Development of a Streamline Curvature Axial-Flow Compressor Performance Simulator Graphical-User-Interface for Design and Research

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ABSTRACT

The all-time interest to increase turbomachinery efficiencies and pressure ratios has led to the progression of more robust and accurate simulation methods and tools. Even though 3-D CFD analyses are highly detailed and despite the computational power nowadays, they can be costly in terms of time and resources. Conversely, 2-D SLC methods provide acceptable performance and flow field results in short times. Because of economical and practical reasons, SLC still represents the cornerstone for turbomachinery design.

In the present, the knowledge demand from the academia community in the airbreathing engine field has been expanding year after year. Nevertheless, there are very few open-source turbomachinery solvers that can be accessed, where user needs to know at least the basics of the programming language syntax and familiarize with it. For these reasons, a GUI was developed for an existing in-house 2-D SLC axial-flow compressor performance code, called SOCRATES. A GUI in this context supports as a teaching mechanism to explain not only the method itself, but also the compressor aerodynamic behaviour.

The SOCRATES GUI consists in the axial-flow compressor model setup, solution and visualization for geometry and results. This paper outlines the main features of the 2-D SLC GUI, and uses a two-stage fan to show the flow field parameters and compressor/fan map, showing a consistent agreement against measured data.

Keywords: Fan; Compressor; Blade Design; Blade Analysis; Through-flow; Streamline Curvature;

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NOMENCLATURE

1-D	One-Dimensional								
2-D	Two-Dimensional								
3-D	Three-Dimensional								
CFD	Computational Fluid Dynamics								
DCC	Dynamic Convergence Control								
IGV	Inlet Guide Vane								
MCA	Multiple-Circular Arc								
NACA	National Advisory Committee for Aeronautics								
OGV	Outlet Guide Vane								
RANS	Reynolds-Averaged Navier-Stokes								
REE	Radial Equilibrium Equation								
SLC	Streamline Curvature								
SOCRATES	Synthesis of Correlations for the Robust Assessment of								
	Turbomachinery Engine Systems								

Symbols

W	Mass flow
x	Radial coordinate in Cartesian system
у	Tangential coordinate in Cartesian system
z.	Axial coordinate in Cartesian system

1.0 INTRODUCTION

The design of turbomachinery components, such as axial-flow compressors and fans still remains an engineering challenge despite of the technology progress. A number of design and analysis tools have been developed since the 1940s [1] to predict performance, every time increasing robustness and accuracy. One of the first approaches to obtain a one-dimensional (1-D) flow field solution relied on a mean-line or pitch-line method developed by Howell [2], where the flow solution is calculated at the blade mid-span in between adjacent blade rows to obtain the overall component performance.

Wu and Wolfenstein [3] first represented the streamline slope and curvature in the radial equilibrium equation (REE), establishing the cornerstone for two-dimensional (2-D) through-flow calculations. Later, Wright and Kovach [4] considered the streamline curvature (SLC) radius in the radial-axial plane to compute flow calculations. With the further expansion of flow simulation through REE solution, Wright and Novak [5] developed one of the first computational codes. Swan [6] developed a computer program where statistics-based empirical viscous and shock loss models were coupled with the REE. Smith [7] properly defined the REE equation for turbomachinery components and similar works in the United Kingdom were presented by Silvester and Hetherington [8]. Nevertheless, a well-defined SLC method for tubomachinery components was firstly defined and introduced [9,10] by Novak [11].

Quasi three-dimensional (3-D) flow analyses were envisaged by Wu [12], where the concept of the two planes: blade-to-blade (S1) and hub-to-tip (S1), was firstly introduced [1]. With the development of computational power in the 1980s, fully 3-D methods were available. In the present, 3-D computational fluid dynamic (CFD) Reynolds-averaged Navier-Stokes (RANS) numerical simulations play a crucial role in the aerodynamic design for turbomachinery components [13]. Turbomachinery 3-D CFD tools numerically solve the viscosity effects at a small scale, however, it is not an exact science [13]. CFD deviations against real parameters can be due to a) numerical errors related to mesh size and finite difference approximation, b) physics modelling assumptions as in turbulence and transition, c) unknown boundary conditions, d) geometry simplification as in the blade leading edge and tip clearance, and e) steady flow assumption [13]. Additionally, CFD simulations come at high computational costs

in terms of solution time and memory, complexity to obtain the required initial and boundary conditions, and lack of flexibility to incorporate or even modify any loss or deviation model [10,14–16]. Under these circumstances, 3-D CFD simulations continue limited to single blade-row models, although there are efforts to conduct multi-stage analyses with the current computational capabilities [17]. Most notable is the fact that in recent years, CFD tools are more widely employed by professionals and young engineers, who despite their expertise, might not realise the CFD drawbacks, representing a potential risk for reliable results procurement.

Alternatively, 2-D through-flow methods provide a quick and fairly acceptable flow solution at low cost in terms of computational run-time and resources [10,15,16,18]. Among the two through-flow techniques known, stream function or matrix method and SLC, the latter is the most widely used [1,19] as it represents the backbone [1,17] for turbomachinery design due to economical and practical reasons [20]. Flow in SLC is assumed to be axisymmetric, compressible, inviscid, and steady. In fact, a fully detailed analysis for an isolated gas-turbine engine component can be obtained through SLC methods. In contrast to CFD, SLC is flexible to incorporate empiricism in the form of loss and deviation models. Besides, SLC numerical simulations require less time to set up the model and the initial and boundary conditions than in 3-D CFD models. Even more, if design optimisation is intended, the 2-D SLC approach avoids the intolerably high 3-D CFD times [20]. Following the progress of 2-D SLC computer programs, several codes have been released over the last 50 years [9,10,19,21–29].

The continuous ambition to increase the compressor efficiencies and pressure ratios has yielded more robust 2-D SLC computational packages. Furthermore, the recent interest of more educators, researchers and students, in the air-breathing engine field has been expanding year after year. Nonetheless, for educational purposes, there are very few open-source tubomachinery solvers available [30]. For instance, turbine codes have been made available, however, user needs to know the commercial package and have a license to run them [31]. Although some compressor and turbine algorithms have been generated as well [32,33], endwall blockage has not been considered. Besides, a comprehensive axisymmetric SLC design system was developed by Turner *et al.* [30], where input files are required to generate the compressor geometry and eventually, obtain the solution.

In this context, a 2-D SLC axial-flow single-stage and multi-stage fan/compressor performance simulator, SOCRATES (Synthesis of Correlations for the Robust Assessment of Turbomachinery Engine Systems), was developed by Pachidis [10], Pachidis *et al.* [14] and Templalexis *et al.* [34], and further improved by Templalexis *et al.* [18] and Templalexis [35]. Templalexis *et al.* [18] explains the SOCRATES structure, where one can find the code-word notation used for the variables and subroutines.

To increase the robustness of SOCRATES, a graphical-user-interface (GUI) was developed for it, motivated by:

- Manual Handling of input and output files opens a window for human errors.
- User needs to familiarize with the input file syntax for compatibility with the code.
- Postprocessing of output file parameters and properties is a time-consuming task that can be automated.
- A GUI for 2-D SLC methods serves as a teaching tool, to understand not only the method itself, but also the concepts and fundamentals for compressor aerodynamic design theory.
- Flexibility in the modification of compressor geometries for blade design.
- Time saving to construct the compressor model and post-process results.
- Feedback from researchers and students that demand more user-friendly tools.

2.0 METHODOLOGY

2.1 2-D SLC Code Structure

The SOCRATES program, coded in FORTRAN 90, follows the 2-D SLC methodology under an iterative technique to re-calculate the streamline position, slope and curvature based on a meridional velocity estimation. Flow field solution is based on the fundamental the Newton's Second Law or conservation of momentum. Because the conservation of momentum already considers the continuity equation, it yields in the Euler equation of motion, which considers the surface traction expressed in terms of the stress field. Due to the inviscid flow assumption, the stress tensor becomes isotropic. resulting in the simplified version of the Navier-Stokes equation for a non-viscous fluid. Within this equation, blade forces are neglected whereas centripetal and Coriolis accelerations are considered. Numerical solution in a cylindrical system, give the full REE to obtain the meridional velocity gradient in the spanwise direction. An initial mesh is generated between the intersection of the assumed initial streamline position, and the inlet and outlet blade rows. REE in set with the mass flow conservation are iteratively solved to satisfy the actual mass flow or outlet static pressure, according to the boundary condition specified. If different, the next loop begins with a new inlet meridional velocity that redefines the streamline radius, and hence, modifying the grid. Streamline radius and shape keeps moving until an agreement is found between the calculated values and specified boundary conditions within a specified error tolerance. Fig. 1 displays a general schematic of the SOCRATES aforementioned processes.



Figure 1 Generic flow chart for the SOCRATES modules.

Due to inviscid flow assumption, empirical correlations are included to compensate for viscosity, deviation and losses. Templalexis *et al.* [18] reported the deviation and loss models included in SOCRATES. Minimum loss incidence angle was calculated with a model from Lieblein [36], while models from Carter [37], Lieblein [38] and Cetin *et al.* [39] were used to calculate deviation angle. Deviation angle at off-design was coded from Creveling and Carmody [21]. Blade row stall prediction was considered from Aungier [40]. Shock Losses were calculated through an empirical correlation that relates a shock loss parameter to the inlet relative Mach number [41].

Besides the flow physics and correlations, the internal iteration algorithms in SOCRATES represent a key feature of the tool. Pachidis *et al.* [42] developed, implemented and tested a dynamic convergence control (DCC) scheme for the solution of the REE in SOCRATES. The new DCC algorithm introduced guarantees

convergence, as in every iteration the error tolerance is tightened at a reasonable solution speed. In a separate study by Templalexis [35], the viscous force terms significance in the flow momentum equation and hence, in the REE, was assessed and introduced in SOCRATES. A closer agreement of the SLC flow field against experimental results was found, when the force term is considered. Despite the increase in the REE complexity with the force term addition, more solutions were converged and fewer iterations were required to achieve convergence.

2.2 Graphical-User-Interface

The SOCRATES GUI was coded in Python v. 3.4.3 and it is divided in three main sections: (1) model setup or pre-processing, (2) solution, and (3) visualization or post-processing, as seen in Fig. 2.



Figure 2 SOCRATES GUI main window workspace.

In general, the model setup is related to the input files, the solution to the program execution, and the post-processing to the output files. Fig. 3 displays a general structure diagram of the GUI, where the different processes for the model setup and post-processing are laid out.



Figure 3 SOCRATES GUI structure for model setup and post-processing.

2.2.1 Model Setup

The first step to characterize the compressor or fan model is to define the number of stages (rotor, stator), inlet guide vane (IGV), outlet guide vane (OGV), or swirler. Additional ducts can be added at the inlet or outlet to capture the flow field properties ahead or behind the active turbo-components as observed in Fig. 4.



Figure 4 Compressor layout model to define the number and type of turbo-components.

The compressor flow path is defined through non-dimensional x and z Cartesian coordinates that allow a potential compressor scaling, based on a maximum flow path length and maximum radius, as illustrated in Fig. 5.

		FlowPathMaxRadius	[cm] 25	.6540		Geomet
owPathRefPoin	tsTip 42		NoOf	FlowPathRefPointsHub 42		
	Flow Path Coord	inates - Tip			Flow Path Coordinate	es - Hub
Inlet 1	-53.3333	100.0000	-	Inlet 1	-53.3333	35.0589
2	-51.3040	100.0000		2	-51.3040	35.0589
3	-44.5347	100.0000		3	-44.5347	35.0589
4	-37.7653	100.0000		4	-37.7653	35.0589
5	-30.9960	100.0000		5	-30.9960	35.0589
6	-24.2267	100.0000		6	-24.2267	35.0589
7	-17.4573	100.0000		7	-17.4573	35.0589
8	-10.6880	100.0000		8	-10.6880	34.7041
9	-3.9147	100.0000		9	-3.9147	35.4448
10	-0.2707	99.9883		10	-0.2707	37.1989
11	2.8587	99.6726		11	2.8587	39.6040
**	6 0.00	07.0402			c 0.000	40.7704

Figure 5 Non-dimensional flow path coordinates definition.

The turbo-component blade row points are specified through non-dimensional x and z Cartesian coordinates. For the case of the rotors and stator, additional information is required, such as number of blades, clearance, and the design performance parameters used for the empirical loss and deviation models, as displayed in Fig. 6.

Lood TE		4. 1st Stage Rotor					
Inlet Duct Inlet Duct Inlet Duct Inlet Duct Ist Stage Rotor		Rotation BladeRov NoOfBlac BladeRov BladeRov	vType les wAxialSpacing [%] vTinClearance [m]	Clocks ROTO 22 21.06/	wise		
. 2nd Stage Rotor . 2nd Stage Rotor . 2nd Stage Stator		BladeRov	vShroudSealFinClearan	ice [m] 0.001	D		
. Outlet Duct . Outlet Duct	RadialPosition	%InRCoord	%OutRCoord	%InZCoord	%OutZCoord	InRelVel	
0. Outlet Duct	Tip 13	100.0000	100.0000	16.1661	41.4310	458.4001	
	12	95.9240	94.9938	15.0310	42.4343	452.7001	
	11	91.5219	89.9954	14.0214	43.2432	445.5001	
	10	82.5861	79.9985	12.6231	44.2779	426.0001	
	9	73.4370	69.9938	11.2748	45.4631	404.7001	
	8	64.1312	59.9969	9.8639	47.0308	383,4001	
	7	54.6059	50.0000	8.3715	48.9496	361.3001	
	(F

Figure 6 Turbo-component blade row definition using non-dimensional coordinates.

To define the blade profiles, blade elements are specified by radial section, which are laid out according to a constant surface turning on a conical surface [43] and stacked along their centre-of-area. Fig. 7 shows the blade-element definition in the SOCRATES GUI.

	NoOfGeometricRefPoints	13				Geometry Defin
Load IF		4. 1st Stage Rotor				
Create IF	RadialPosition	Camberline	ThickDistr	StaggerAng	InletAng	TransitionAng
. 1st Stage Rotor	Tip 13	PA 🔻	3rdPoly 👻	-63.8600	-66.6100	-64.8300
. 1st Stage Stator	12	PA	3rdPoly 👻	-61.5900	-64.5600	-62.3900
. 2nd Stage Stator	11	PA -	3rdPoly -	-59.6300	-62.8300	-60.1400
	10	PA	3rdPoly -	-56.9400	-60.8500	-56.8600
	9	PA	3rdPoly 👻	-54.0100	-59.0100	-53.9800
	8	PA	3rdPoly -	-50.3500	-56.8100	-50.8800
	7	PA -	3rdPoly 👻	-45.8400	-54.2700	-47.4000
	6	PA	MCA -	-40.4500	-51.4000	-43.4700
	5	PA -	MCA -	-33.6500	-47.4400	-39.0400
	4	PA	3rdPoly -	-25.9800	-43.7900	-34.9700
	3	PA	MCA -	-18.0200	-41.4000	-31.8100
	2	PA	MCA -	-14.0200	-40.3000	-30.5900
	٠		·			F.

Figure 7 Blade-profile-element definition for different radial positions.

Because the solution of the REE is an interative approach, initialization values for the mean-line meridional velocities at the turbo-component inlets and outlets are specified. Additionally, endwall blockage factors can be indicated at the blade row inlets and outlets. Mean-line meridional velocities and blockage factors are given as in Fig. 8.

1. Inlet Duct 2. Inlet Duct 3. Inlet Duct 4. 1st Stage Rotor 5. 1st Stage Statur			
6. 2nd Stage Rotor 7. 2nd Stage Stator 8. Outlet Duct	Air	irInletTipBlockageFactor irInletHubBlockageFactor	0.0000 4.0000
9. Outlet Duct 10. Outlet Duct	Air	irOutletTipBlockageFactor irOutletHubBlockageFactor	0.2000
	Air	irInletMeridMeanLineVel	185.4000
		Clear	ave

Figure 8 Initialization values and blockage factors specification.

In terms of the design and off-design cases to analyse, different speedline points can be specified where particular boundary conditions can be established for them as depicted in Fig. 9. As mentioned in Sec. 1, either inlet mass flow or outlet static pressure can be used as boundary condition.

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F	low Field Prope	rty Profiles at Co	mpressor Inlet			
Load IF N	oOfBoundaryPoints	13	NoOfTimeSteps	10	RotationalSpeedDP [rpm]	16042.8
Create IF	2	Time Step 4				
		ExperReading			1382	
		ThetaCoord o	r TimeStep		4.0000	
		BoundaryCon	ditionSwitch		MassFlowInlet -	
		InletMassFlov	v		34.2660	
		RotationalSpe	ed [rpm]		16090.0000	
	RadialPosition	%InRCoord	AbsTotTemp	AbsTotPres	MeridAbsAn	g
	Tip 13	100.0000	289.2000	9.8200	1.5000	
)	12	95.3481	289.0000	9.9100	0.9000	
	11	91.1405	288.8000	10.1000	0.5000	
	10	82.5870	288.6000	10.1500	0.5000	
	9	73.8235	288.1000	10.1600	0.4000	
	8	64.9160	288.1000	10.1600	0.4000	
	7	55.8043	288.1000	10.1600	0.0000	

Figure 9 Boundary conditions for every speedline operating point specified.

To finalize the model setup, solution settings are specified in terms of the number of streamlines for the grid, damping factor for streamline radius movement between iterations, and the different error tolerances to satisfy boundary conditions.

NoOfStreamlines	31	[Use only odd numbers]
NoOfBladeChordLocations	26	
NoOfBladeAxialPoints	50	
StrlineMovemDampingFactor	-12	
OutletStatPressErrorToler	0.0010000000	
InletMassFlowErrorToler	0.0010000000	
PassageChokeMarginCriter	1.5	
	Clear Save	

Figure 10 Solution settings input.

2.2.2 Solution

The SOCRATES execution can be performed directly in the GUI or through a quick launching tool developed that allows a selection of the different compressor/fan geometries available. The quick launcher allows GUI access in case a model is modified or a new compressor is defined.

SOCRATE	S QUICK Launch		_		×
Socrates	Plotting Settings				
NASA TP 1	493 100 🔻		New Comp	ressor	
Run		Add			
🗹 Enable p					
Number of i	800			-	
Start		Cance	el		

Figure 11 Quick launcher to directly run SOCRATES or start the GUI.

A vast library or compressor and fan geometries have been modelled over the development years of SOCRATES. Currently, the following geometries are available in SOCRATES: NASA Two Stage Fan [44], NASA Rotor 67 [45], NASA Rotor 66 [46], NASA Rotor 37 [47], NASA ADP Fan [48], NASA QF-1 Fan [49] [50], NASA Compressor 74A [51], NASA Two-Stage Fan with Dampers[52] and the NASA Stage

38 [53]. In this paper, the NASA Two-Stage Fan [44] is used as instance to display the different GUI utilities.

During running time, the residuals for the steamtube inlet and outlet mass flow errors for each turbo-component are plotted to track convergence. Fig. 12 shows the residuals graphs, which are constantly updated at every iteration.



Figure 12 Iteration residuals plots for the inlet and oulet of each turbo-component.

Once the computation converges, output files for geometry, flow field parameters and performance characteristics are generated.

2.2.3 Visualization

The compressor flowpath sketch is visualized in a 2-D meridional plane, where the defined turbo-components are laid out as seen in Fig. 13.



Figure 13 Axial-flow compressor 2-D view in the meridional plane.

Similarly, a blade-to-blade view can be obtained for every blade radial position assigned to appreciate the profile, as observed in Fig. 14. Further, a tabulation for the blade profile Cartesian coordinates in the x, y and z –axis is provided.

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4. 1st Stage Rotor	Radia	positi	on selector 4. 1st Stage R	otor	
6. 2nd Stage Rotor		[GeometricRefPoint 7	D W	isualisation
7. 2nd Stage Stator		Axial Point	Camberline Z1-COORD	Camberine Y1-COORD	Dist window
lade selectio	on list	1	1.3294	3.7544	Allows to vieweling the blade coefic is a two dimensional
	2	2	1.4614	3.5762	⁴ plane, according to the radial position selected
	3	3	1.5933	3.4004	
	4	4	1.7252	3.2269	
	5	5	1.8571	3.0558	
	6	6	1.9891	2.8869	2
	7	7	2.121	2.7202	5
	8	8	Coordinates tak	2.5557	ixe I
	9	9	2.3849	2.3934	et o
	10	10	2.5168	2.2331	Gue
	11	11	2.6487	2.0749	
	12	12	2.7807	1.9187	-2
	13	13	2.9126	1.7645	
	14	14	3.0445	1.6123	
	15	15	3.1765	1.462	
	16	16	3.3084	1.3135	-4
	17	17	3.4403	1.167	0 2 4 6 8
	18	18	3.5722	1.0223	Visulaization toolbar
	19	19	3.7042	0.8793	A A A 1
					1 V V T V EI m V

Figure 14 Blade-profile-section 2-D view in the blade-to-blade plane.

Due to the blade-element layout method implemented, full 3-D coordinates are obtained for the whole compressor/fan geometry. The 3-D view of the compressor flowpath and blading can be visualized as depicted in Fig. 15.



Figure 15 3-D visualization for the NASA two-stage fan [44].

The final mesh established between the converged streamlines and their corresponding quasi-orthogonals can be visualized as in Fig. 16. A more detailed zoom into the grid allows identifying the streamline displacement at the endwalls due to the blockage factors as shown in Fig. 17.

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Figure 16 2-D final grid in the meridional plane composed by converged streamlines and quasiorthogonals.



Figure 17 Computaional 2-D grid visualization zoom at hub endwall.

3.0 RESULTS

To show the SOCRATES GUI results post-processing, the NASA Two-Stage Fan [44] was modelled, simulated and used to illustrate this section. Table 1 lists the design overall parameters for this two-stage fan.

Table 1 NASA Two-Stage Fan [44] design overall parameters

Parameter	
Rotational Speed [rpm]	160428.8000
Fan Total Pressure Ratio	2.3990
Fan Total Temperature Ratio	1.3340
Fan Adiabatic Efficiency	0.8490
Mass Flow [kg/s]	33.2480
First-Stage Tip Speed [m/s]	428.8960

3.1 Flow Field Parameters

A post-processing plotting tool was developed to plot the flow field properties at each turbo-component blade row station. For direct response comparison between the same or different turbo-component inlet and outlet blade rows, several curves can be plotted in the same graph as seen in Fig. 18. Typically, spanwise properties distribution is desired; however, the post-processing tool allows modifying the variables in both graph axes.



Figure 18 Flow field properties plotting

Fig. 19 shows the potential of the tool to compare different properties. In this case, the 1^{st} stage rotor inlet against the outlet behaviour at w = 34.515 kg/s and design speed is compared.



Figure 19 2-D SLC NASA Two-Stage Fan [44] flow field parameters for the 1st stage rotor inlet and outlet. a) Absolute total temperature b) Absolute total pressure c) Static pressure d) Relative meridional flow angle e) Relative Meridional Mach number f) Meridional Velocity

Apart from flow field parameters, blade-profile elements can also be plotted at every streamline radial location. Fig. 20 displays the aero-chord spanwise distribution.



To validate the 2-D SLC flow field from the GUI, Fig. 21 shows a comparison against measured data, where consistent agreement is obtained and no significant differences are observed.



Figure 21 NASA Two-Stage Fan [44] flow field parameters comparison between experimental data and 2-D SLC for the 1st stage rotor inlet and outlet. a) Absolute total temperature b) Absolute total pressure c) Static pressure d) Relative meridional flow angle e) Relative Meridional Mach number f) Meridional Velocity

3.2 Performance Characteristics

A different workspace is used by the post-processing tool to plot the performance map for different speedlines analysed. The number of operating points appearing on the map depends on the number of cases specified in the boundary conditions. Fig. 22 shows the pressure ratio fan map for the NASA Two-Stage Fan, whereas in Fig. 23, the isentropic efficiency fan map is shown.



Figure 22 Fan pressure ratio map for the NASA two-stage fan [44] at 100, 90, 80, 70 and 50% of design speed.



Figure 23 Fan isentropic efficiency map for the NASA two-stage fan [44] at 100, 90, 80, 70 and 50% of design speed.

The plotted 2-D SLC performance characteristics are compared against measured results in Fig. 24 for the pressure ratio and Fig. 25 for the isentropic efficiency. A satisfactory agreement is obtained for the pressure ratio. For the isentropic efficiency, although there is a difference between the experimental and simulated curves, there is a qualitative trend agreement. Difference in isentropic efficiency is less than 5% between the experimental and 2-D SLC data at the peak-efficiency points of each speedline. The difference is explained due to the empirical profile-loss models, which can be further fine-tuned to match the efficiency.



Figure 24 NASA Two-Stage Fan [44] pressure ratio map comparison between measured and 2-D SLC data.



Figure 25 NASA Two-Stage Fan [44] isentropic efficiency map comparison between measured and 2-D SLC data.

4.0 CONCLUSIONS

The demand for more robust and accurate turbomachinery flow simulation methods and tools has expanded in the recent years. Unlike 3-D CFD analyses, 2-D SLC methods offer an acceptable solution in minutes. Because of the lack of open-source turbomachinery codes and the learning curve that user needs to go through, a GUI was developed for SOCRATES, an existing in-house 2-D SLC axial-flow compressor simulator. A GUI in this context helps to understand the 2-D SLC method itself and the axial-flow compressor aerodynamics, apart from saving time in the model preparation and results post-processing.

The SOCRATES GUI was built in three main sections: model setup, solution and visualization. The model setup handles the input files to define the compressor geometry, boundary conditions, initialization values, and solution settings. The solution module consists in a quick launcher to execute the simulation and mass flow residuals

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plotting during running time. In the visualization module, the axial-flow compressor is displayed in the 2-D meridional plane and in a 3-D view, along with a blade-to-blade projection for every blade-profile section. Furthermore, the visualization allows to post-process the result output files for the flow field properties at every turbo-component inlet and outlet, and for the compressor/fan performance map. Over the development years of SOCRATES, several axial-flow compressor geometries have been modelled. Among these, a NASA two-stage fan was used to validate the results obtained from GUI the post-processing. 2-D SLC flow field parameters and overall pressure ratio proved to be matched against experimental results. In terms of the isentropic efficiency map, differences less than 5% at the speedline peak-efficiencies were observed induced by the empirical loss models, nonetheless, the trend between 2-D SLC and measured data was consistent.

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