EXPERIMENTAL STUDY OF HYDRAULIC JUMP CHARACTERISTICS IN TRIANGULAR CHANNEL

ETUDE EXPERIMENTALE DES CARACTERISTIQUES DU RESSAUT HYDRAULIQUE DANS UN CANAL TRIANGULAIRE

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ABSTRACT

The hydraulic jump is the turbulent transition from supercritical to subcritical flow. The aim of this work is to contribute to a best understanding of the behavior of hydraulic jump in a symmetrical triangular channel of central angle of 90°. New experiments were conducted to quantify its characteristics for a wide range of Froude numbers (2 < F1 < 11). The analysis of the experimental results shows the insufficiency of the equations already suggested, deducted from the half triangular channel studied by *Hager and Wanoschek* (1985). The influence of the upstream sequent depth on the features of the hydraulic jump and especially the surface profile is considered.

Some relationships, translating the sizes of the characteristics of the hydraulic jump in asymmetrical triangular channel, are presented in dimensionless terms, in order to show their general practical validity. The effects of the cross-sectional are investigated as well.

This research is applied particularly in the conception of dams' stilling basins.

KEYWORDS: Hydraulic jump; stilling basin; triangular channel; classical jump; Energy dissipation.

RESUME

L'expérimentation du ressaut hydraulique classique dans un canal triangulaire symétrique à angle d'ouverture de 90° a permis une meilleure compréhension de son comportement. Les analyses des mesures expérimentales ont montré les insuffisances des relations jusqu'alors proposées, déduites du profil semi-triangulaire étudié par *Hager et Wanoschek* (1985). Des relations traduisant les grandeurs des caractéristiques du ressaut hydraulique dans un canal triangulaire symétrique sont présentées en termes adimensionnels pour que leurs validités pratiques soient générales. L'influence de la hauteur initiale sur les différentes caractéristiques du ressaut, notamment le profil de surface, est mis en évidence. Cette recherche trouve son application dans la conception des bassins amortisseurs des barrages.

MOTS CLES : Ressaut hydraulique ; canal triangulaire ; dissipateur d'énergie ; bassin d'amortissement; ressaut classique.

1 INTRODUCTION

Many hydraulic structures such as surface spillways and shaft spillways are designed to remove water and restore it further downstream of a river. In most cases this removal of water results in the conversion of the potential energy stored in a reservoir into high kinetic energy downstream of the inflow stream release structure. The driving forces generated, proportional to the square of the velocity, can seriously threaten the structure due to their highly erosive nature. Therefore it is often necessary to dissipate the greater part of this kinetic energy.

The flow at the entry of the inflow stream release structure is in supercritical regime, characterised by a Froude number higher than one. The principle of dissipation consists in transforming this supercritical regime into a subcritical river regime, characterised by a relatively low velocities downstream of the energy dissipater. A hydraulic jump is thus formed throughout the length of the stilling basin, often of rectangular shape and sometimes equipped with baffles, sills, etc. positioned width-wise. The main characteristics of a classical or forced jump are the efficiency, representing the head loss ratio caused by the initial load in its upstream section, the length and upstream and downstream sequent depths.

According to *Debabeche et al* (2006), *Debabeche et al* (2011) and *Chanson* (2012), from the quantitative view point, the characteristics of the jump mainly depend on the geometry of the stilling basin, the inflow Froude number and the relative sill height s/h1for the case of the forced jump, with s and h1 representing the geometric height of the sill and the upstream depth of the jump.

For the sake of economy and efficiency, the jump length as well as the downstream depth must be as short as possible, and dissipating the maximum amount of energy. Until now, no theoretical development has been carried out to established a relationship making it possible to the evaluation of the roller length Lr and the jump length Lj; this has only been achieved through experimental tests (*Bessaih et al., 2002; Abdul, 2008; Wang et al., 2015*). Sanjeev et al. (2010) and Richard et al. (2013) have proposed non-dimensional relations to calculate the relative length $\lambda_{\rm r}$.

The works of *Sanjeev et al.* (2010) showed that the length of the jump is as long as M tends to infinity. The increasing values of M lead to a trapezoidal profile geometrically shrunk at the base, which gives rise to a more a triangular profile.

However, jumps placed in channels with triangular crosssections have not attracted as much interest as those in rectangular cross-sections (*Abdul, 2008*; *Chyan-Deng et al., 2009*; *Sanjeev et al., 2010*; *Richard et al., 2013*), although it appears that the triangular profile is better adapted to the role of dissipation.

The first authors to have studied this type of jump were Argyropoulos and Rajaratnam, by analysing the results obtained by Silvester. The former carried out his experiment in a channel with aperture angle of 47.3°, whereas the latter tested a channel with an aperture angle of 60° . The results showed that, overall, the equation of the quantity of movement is generally verified although certain measurement points tend to show that the sequent depth ratios Y are slightly lower than the theoretical values. This variance, estimated at 5%, was also observed in the analysis of the results obtained by (Hager et al., 1985) for a triangular channel with an aperture angle of 90° (m = 1). A jump controlled by a weir was analysed experimentally by (Achour et al., 2003; Debabeche et al., 2006) and the effect of the weir on the compactness of the stilling basin was observed and quantified.

Mention must be made of the specific approach taken by (*Hager et al.*, 1985) in a jump experiment. A wall inclined at 45° (m = 1) y was positioned in a channel with a rectangular cross-section. The geometrical profile obtained and on which the tests were performed was semi-triangular, formed by a vertical wall and in inclined wall. The experimental results obtained using this profile, were extrapolated to a symmetrical triangular profile. The

reliability of this extrapolation may appear poor with respect to the influence of the vertical wall on the characteristics and behaviour of the hydraulic jump.

In this context, the present study first presents an experimental examination of the behaviour and the characteristics of hydraulic jump placed in a channel with a symmetrical cross-section with an aperture angle of 90°. The series of measurements performed by varying the initial depth of the jump (h_1 = constant for each series of tests) allowed us to observe the influence of the upstream depth h_1 on the jump characteristics, especially the profile of the axial surface. Secondly, the experimental results obtained are compared with those obtained by the semi-triangular cross-section channel jump, and with the theoretical development.

2 MATERIALS AND EXPERIMENTAL PROCEDURE

The tests were performed in an experimental set-up (Figure 1), composed by a stilling basin linked to a filling tank with a capacity of 2000 litters, by the mean of a pipe with 110 mm of diameter equipped with a horizontal pump with a flow rate ranged from 0 to 30 l/s, a horizontal channel 4 meters long, sufficient for the full formation of a jump for the entire range of flow values Q (4 l/s \leq Q (l/s) \leq 30). The flow values were measured using an orifice type velocity meter with an accuracy of \pm 0.5 l/s. The installation of a convergent nozzle at the outlet of the filling tank generated a high velocity flow. A sill was positioned at the downstream of the channel to return the jump to the upstream preferential position.

For each upstream depth h_1 , the final depth h_2 was measured, as well as the length L_r of the roller, the length L_j of the jump, the flow rate Q and the heights of the free surface water stream.

The tests were performed with the upstream sequent depths h_1 which corresponds to the opening of the C nozzle outlet such as $38 \le h_1$ (mm) ≤ 79 and with a Froude number ranging from 2 to 11.

For each of the values of depth h_1 measurements were made of the flow rate Q, the sequent depth downstream h_2 , using a dual scale staff gauge accurate to ± 1 to 1.5 cm, due to the considerable fluctuations of the free surface flow downstream. The horizontal distances, in particular the length of roller L_r and the length of jumper L_j , were measured using a graduated strip fixed along the wall of the channel. To obtain the profile of the free surface flow of the water in the centre of the channel, about ten staff gauges were installed on it to measure the water depths along the whole length of the jump.



Figure 01: Schematic layout of experimental setup

3 RESULTS AND DISCUSSION

3.1 Variation of sequent depth ratio with approach Froude number

The experimental verification of the relationship giving the sequent depth ratio Y resulting from the application of the momentum equation was necessary. Firstly, to confirm experimentally the validity of the equation of the quantity of movement momentum equation expressing the ratio Y of the sequent depths h_1 and h_2 as a function of the Froude number F_1 . It should be noted that the theoretical ratio Y for a triangular channel is given by the following relation Hager et al. (1985):

$$F_1^2 = \frac{2Y^2(Y^2+Y+1)}{3(Y+1)}$$

Secondly, this was done to verify the effect of friction of the channel bed and walls on ratio Y and confirm that this ratio did not depend on the Froude number F_1 .

Curve $Y = f(F_1)$ of figure 2, resulting from the experiment shows that the influence of the head loss on Y ratio is negligible, or non-existent. The coincidence of the experimental points with the theoretical curve of equation 1 indicates the absence of the effect of friction on the sequent depth ratio throughout the range of F_1 tested, i.e. 2 $\leq F_1 \leq 11.$ We can confirm the validity of the theoretical relation (1) for the classical jump in a triangular channel with an

aperture of 90° and invalidate the consideration of the effect of friction expressed by Hager et al. (1985).



We propose to express ratio Y_T of the jump moving in a

basin with a triangular cross-section as a function of ratio YR given by the equation of Belanger :

:
$$Y_R = \frac{\sqrt{1+8F_1^2}-1}{2}$$

This ratio is presented by the following relation:

$$Y_{\rm T} = 0.9 Y_{\rm R}^{2/3} + 0.1$$

Compared to the approximated solution proposed by Hager et al, (1985),

$$Y = \sqrt{2F_1} - \frac{1}{2} for M=0$$

And

$$Y = \left(\frac{3F_1^2}{2} - 1\right)^{1/3}$$
 1/M=0
(3)

The relative error is higher than 1%.Relation (2) presents a simpler approximated solution with better precision as its relative error does not exceed 0.3% (Figure 3). For a Froude number $F_1=1$, the value of Y_T according to equation (2) is equal to one, which moreover correctly expresses the critical phase of the flow.

However, for $F_1=1$, relation (3) gives a ratio $Y \neq 1$.



Figure 03: Theoretical and experimental variation of Y

3.2 Characteristic jump lengths

3.2.1 Relative length of Roller

The lengths of both the roller and jump are difficult to measure due to the instability of the jump and the quite considerable imprecision with which the corresponding sections were localised experimentally. We have deliberately represented the variation of the relative length of the roller L_r/h_1 as a function of Y-1 instead of F₁ as is usually done. The main reason is that this representation permits eliminating errors made on the flow rate measurements and thus on the calculation of the Froude numbers F1. Figure 4 shows the experimental data resulting from seven series of measurements performed under upstream depth h1 and varying between 45 and 79 mm. Thus, we observed a cloud of points distributed between two enveloping curves defined as:

$$6.0(Y-1)^{1.3} \le \frac{L_T}{h_1} \le 8.25(Y-1)^{1.21}$$

(4)

 $45 \text{ mm} \le h_1 \le 79 \text{ mm}$



Figure 04: Variation of relative length of roller as a function of Y-1. (+ + +) Experimental data,

45 mm \leq h₁ \leq 79 mm

The error usually accepted in the measurement of characteristic jump lengths is ± 20 cm; the uniform distribution of the data is sufficiently significant. For the values Y-1>1, i.e. Y>2 (a value greatly exceeded in practice), we can assume that the relative lengths L_r/h_1vary around a mean curve with a practically linear trend. This line is located between the envelope curves defined by relation (4) and has the following equation:

$$\frac{L_{\rm T}}{h_1} = 11 \, ({\rm Y} - 1) - 4$$
 Y > 2
(5)

3.2.2 Relative length of Jump

This length, which is difficult to measure, especially for quite high upstream Froude numbers ($F_1>5$), is shown in Figure 5 on the basis of experimental values of lengths relating to jump L_j/h_2 as a function of the Froude number F_1 . This graph clearly shows that the mean experimental curve for the jump is located far above the curve plotted

according to the following equation Hager et al. (1985):





Figure 05: Variation of relative length of jump as a function of F_{1} . (--) Equation 6, (x x x) Experimental data for

45 $mm \le h_1 \le 79 mm$

This considerable variance between the two curves, first expresses the effect of the vertical wall of the semitriangular channel on the length of the jump and, second, the non-validity of equation (6) for the case of the symmetrical triangular jump. The lengths for jump L_i/h₂vary around a mean curve whose values can reach 9 for Froude numbers between 6 and 8.

3.3 Generalised surface profile

Variables
$$X = \frac{x}{L_j}$$
 and $Y = \frac{h(x) - h_1}{h_2 - h_1}$

Plot the surface profile of the jump in non-dimensional form.

The relation governing the generalised non dimensional profile y(x) along the axis of a non-symmetric triangular channel for several Froude numbers F₁ obtained by Hager et al. (1985) is written as follows:

$$y = (1 + \propto X) \tanh(\beta X)$$

For $0 \le X \le 0.2$ the surface profile has an approximately linear slope $\beta = dy/dx = 2.5$

In figure 6 it is presented the curve resulting from relation (7) for the case of a semi-triangular jump with h_1 values varying between 70 and 76 mm and the measurement points corresponding to the h1values between 45 and 79 mm. It was able to observe that for 76 mm \leq h₁ \leq 79 mm the experimental points of the symmetrical triangular jump were located around the curve of equation (7). However, for the values of $h_1 < 76$ mm, the distance between the

curves obtained increased progressively while h_1 decreased. This indicated that relation (7) can only be applied to values of $h_1 \ge 76$ mm and shows the influence of h₁ on the shape of the generalised surface profile of the triangular jump (m = 1).



Figure 06: Evolution of generalised non dimensional profile

y(x) along the axis of channel (--) Equation 7, (xxx) Experimental data for

 $45 mm \le h_1 \le 79 mm$

3.4 Jump efficiency

Theoretically, for $F_1 > 3$ the efficiency of the triangular jump is higher than that of the rectangular one; the variance can reach 10% above $F_1 = 5$. The efficiency is then evaluated approximately by relation (8):

」 1/M = 0

(8)

$$\eta_{\rm R} = \left(1 - \frac{\sqrt{2}}{F_1}\right)^2_{M=0}$$
$$\eta_{\rm T} = \left[1 - \left(\frac{12}{F_1^4}\right)^{1/3}\right]^2_{M=0}$$

CONCLUSIONS 4

The main characteristics of the classical jump moving in a symmetrical triangular channel with an aperture of 90° were evaluated experimentally. The installation of a convergent nozzle under pressure generated a high velocity flow and made it possible to avoid measuring the upstream depthh₁. The analysis of the experimental values of sequent depths h₁and h2showed that the effect of friction on ratio $Y = h_2 / h_1$ was negligible and that the equation of the quantity of movement was verified. An approximate relation of ratio Y of the symmetrical triangular jump was proposed. In addition, the measures of the relative lengths of roller Lr/h1 for 45 mm $\leq h_1 \leq 79$ mm led to the establishment of a mean curve with a practically linear situated between two envelope curves. The relative length of the jump L_i/h_2 could reach 9 for 6 mm $\leq h_1 \leq 8$ mm, whereas for the same range of F_1 , the value of L_i/h_2 was in the region of 5 in the semi-triangular channel. The influence of the initial depth h1on the shape of the surface profile of the jump was observed. The application of the equation proposed up to then remains limited to the values $h_1 \ge 76$ mm. The experimental points of the surface profile progressively tended away from the curve already established as h₁ decreased. The maximum efficiency of the rectangular jump was about 71%, whereas it could reach 73% for the triangular jump, i.e. 2% more for the same Froude number $F_1 \cong 9$. The study showed that the symmetrical triangular jump was characterised by a very different behaviour compared to the jump in the semitriangular channel. This difference stemmed from the symmetry of the cross-section of the jump which permitted performing measures along the centre of the channel, which was not possible in the semi-triangular section; however, it also stemmed from the difference in the quantitative and qualitative characteristics of the jump.

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Nomenclature

λ _i (-)	Relative jump length, λ	$_{i} = L_{i}/h_{2}$
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- λ_r (-) Relative roller length, $\lambda r = L_r/h_1$
- F₁ (-) Inflow Froude number
- H (m) Total load
- h (m) Flow depth
- h_1 (m) upstream Sequent depth
- h_2 (m) downstream Sequent depth
- $L_r(m)$ Length of roller
- $L_j(m)$ Length of jump

m (-) Cotangent of the angle of inclination of the wall with regard to the horizontal

M (-) Relative upstream depth of jump,

M = mh1/b

Q (m3s-1)	flow discharge
X (-)	Non-dimensional longitudinal coordinate,
	X = x/Lj
x (m)	Longitudinal coordinate
Y (-)	Sequent depth ratio
YR (-)	Sequent depth ratio in rectangular
	channel
YT (-)	Sequent depth ratio in triangular channel
у (-)	Non-dimensional vertical coordinate,
	$y = \frac{h(x) - h_1}{h_2 - h_1}$
α(-)	Correction coefficient, for $X = 1$ and
	$y = 1, \alpha = 0.014$
η (-)	Efficiency of jump
ηR (-)	Efficiency of jump in rectangular
	channel
ηΤ (-)	Efficiency of jump in triangular channel