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VALIDATION OF COMPUTATIONAL FLUID DYNAMIC SIMULATIONS
OF MEMBRANE ARTIFICIAL LUNGS WITH X-RAY IMAGING

DISSERTATION

A dissertation submitted in partial fulfillment of the requirements
for the degree of Doctor of Philosophy in
Biomedical Engineering at the University of Kentucky

By

Cameron Christopher Jones

Lexington, Kentucky

Director: Dr. Joseph B. Zwischenberger, Johnson-Wright
Professor and Chairman, Department of Surgery

Lexington, Kentucky

2012

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ABSTRACT OF DISSERTATION

VALIDATION OF COMPUTATIONAL FLUID DYNAMIC SIMULATIONS OF MEMBRANE ARTIFICIAL LUNGS WITH X-RAY IMAGING

The functional performance of membrane oxygenators is directly related to the perfusion dynamics of blood flow through the fiber bundle. Non-uniform flow and design characteristics can limit gas exchange efficiency and influence susceptibility of thrombus development in the fiber membrane. Computational fluid dynamics (CFD) is a powerful tool for predicting properties of the flow field based on prescribed geometrical domains and boundary conditions. Validation of numerical results in membrane oxygenators has been predominantly based on experimental pressure measurements with little emphasis placed on confirmation of the velocity fields due to opacity of the fiber membrane and limitations of optical velocimetric methods.

A novel approach was developed using biplane X-ray digital subtraction angiography to visualize flow through a commercial membrane artificial lung at 1–4.5 L/min. Permeability based on the coefficients of the Ergun equation, α and β , were experimentally determined to be 180 and 2.4, respectively, and the equivalent spherical diameter was shown to be approximately equal to the outer fiber diameter. For all flow rates tested, biplane image projections revealed non-uniform radial perfusion through the annular fiber bundle, yet without flow bias due to the axisymmetric position of the outlet. At 1 L/min, approximately 78.2% of the outward velocity component was in the radial (horizontal) plane versus 92.0% at 4.5 L/min. The CFD studies were unable to predict the non-radial component of the outward perfusion.

Two-dimensional velocity fields were generated from the radiographs using a cross-correlation tracking algorithm and compared with analogous image planes from the CFD simulations. Velocities in the non-porous regions differed by an average of 11% versus the experimental values, but simulated velocities in the fiber bundle were on average 44% lower than experimental. A corrective factor reduced the average error differences in the porous medium to 6%. Finally, biplane image pairs were reconstructed to show 3-D transient perfusion through the device.

The methods developed from this research provide tools for more accurate assessments of fluid flow through membrane oxygenators. By identifying non-invasive techniques to allow direct analysis of numerical and experimental velocity fields, researchers can better evaluate device performance of new prototype designs.

Keywords: Artificial Lung, X-ray Angiography, Computational Fluid Dynamics, Porous Media, Experimental Validation Methods

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April 27, 2012

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VALIDATION OF COMPUTATIONAL FLUID DYNAMIC SIMULATIONS
OF MEMBRANE ARTIFICIAL LUNGS WITH X-RAY IMAGING

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April 27, 2012

DEDICATION

I dedicate this body of work to my parents, Donald Jones and Cindy Vnencak-Jones, PhD,
for their continued support, encouragement, and love;
and to
my sister, Kelly; may God inspire you and guide you through all of life's journeys.

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1 Thessalonians 5:16–18.

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1. INTRODUCTION

1.1. *Statement of Contribution*

It is the purpose of this dissertation to identify a method for obtaining experimental perfusion data of blood flow through a membrane artificial lung in order to verify the accuracy of numerical models. A novel approach was developed using X-ray angiography to visualize flow through a commercial membrane oxygenator with real-time, high spatio-temporal resolution. Two-dimensional velocity fields in the fiber bundle were generated from a pattern-searching tracking algorithm, allowing evaluation of the values obtained from numerical simulations.

Permeability through the fiber bundle was compared with empirical relationships of flow through porous media, and the results were directly applied to formulations in the macroscopic computational fluid dynamic (CFD) analysis of blood flow through membrane artificial lungs. With the insight gained from the experimental measurements, a correction factor was defined which improved the post-processing results of the CFD models. In addition, a new 3-D reconstruction technique improved clarity of the transient fiber bundle perfusion, which is useful for assessing uniformity of the flow distribution.

The results of this research show that conventional methods for modeling permeability through fiber membranes greatly under-predict the true physical velocity of the blood flow, and emphasize the need for experimental validation of CFD simulations through porous media. The methods developed herein will provide tools for more accurate assessments of fluid flow through membrane oxygenators. By identifying non-invasive techniques to allow direct analysis of numerical and experimental velocity fields, researchers can better evaluate device performance of new prototype designs, which may improve the outcome from extended use of membrane oxygenators.

1.2. *Structure of Dissertation*

The contents of this dissertation are partitioned into chapters according to the traditional format: Background [ch. 2]; Methods [ch. 3–4]; Results [ch. 5–6]; Discussion [ch. 7]; and Conclusions [ch. 8]. Each chapter is further divided into subsections (§) for clarity, while most sections and major themes are followed by a brief discussion to revisit applications, limitations, and concepts from the preceding text. Abbreviations and notations are defined when initially presented and are available for easy reference in the Appendix section at the end of the dissertation.

2. BACKGROUND AND LITERATURE REVIEW

2.1. Extracorporeal Membrane Oxygenation

Extracorporeal membrane oxygenation (ECMO) therapy can provide prolonged functional support for patients with severe cardiac and/or pulmonary failure. Effectively a modified heart-lung machine used in cardiopulmonary bypass (CPB) procedures, the principles behind ECMO have been used in applications of whole-body hyperthermia¹, CO₂ removal^{2,3}, and total artificial lungs.⁴ Though the applications of extracorporeal circulation are widespread, the focus of this research is on long-term respiratory support.

2.1.1. Respiratory Failure

Chronic respiratory diseases are the fourth leading cause of death in the United States and include conditions such as chronic obstructed pulmonary disease (COPD), emphysema, and other lower respiratory diseases.⁵ While lung transplantation is the most effective treatment for end-stage pulmonary diseases, there are currently over 1700 patients on the waiting list, with nearly 18% wait-list mortality.^{6,7} The demand for long-term pulmonary support exists also for patients with reversible respiratory diseases such as pneumonia and acute respiratory distress syndrome (ARDS).

2.1.2. Long-Term Pulmonary Support

Patients requiring prolonged, but temporary, respiratory support are most often treated with mechanical ventilation, among other therapies such as inhaled nitric oxide, surfactant, and prone positioning.⁸ But progressive mechanical ventilation can cause additional lung injury

(volutrauma), infectious complications from long-term tracheal access, and only allows partial lung support—limited by the gas exchange capabilities of the remaining lung parenchyma.^{9,10}

In cases of severe, acute respiratory failure (such as ARDS), ECMO may be used to provide temporary support (days–weeks) for lung recovery, while avoiding the potential for secondary injury resulting from mechanical ventilation. Unfortunately, ECMO is usually only considered as a last resort due to the complexity of the circuit, imparted blood trauma, and the daily cost of the therapy. Current patient management protocols in ECMO support can provide extracorporeal circulation for 1–6 weeks, but the median duration is about 1 week.⁸

2.1.3. ECMO Circuit

The fundamental components of the ECMO circuit are shown in Figure 2.1. Venous (deoxygenated) blood is drained from the patient allowing carbon dioxide (CO₂) removal and blood oxygenation to occur via an external device, and is returned to the patient’s venous or arterial circulation. Extracorporeal access can be configured for either single (with the use of a double lumen catheter) or multiple cannulation sites, depending on the patient needs.

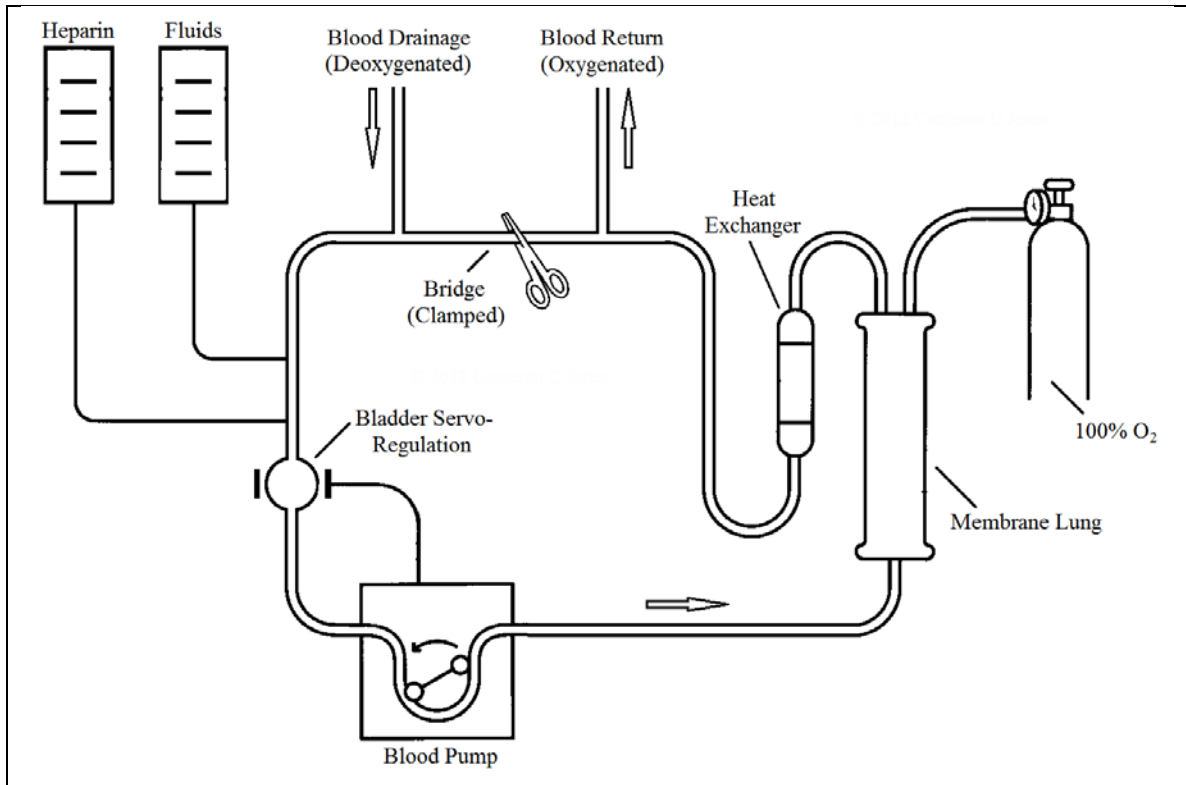


Figure 2.1: ECMO Circuit

The principle function of the extracorporeal circulation is to perform all necessary gas exchange for normal physiologic health of the patient.

The complexity of the ECMO circuit introduces several challenges to long-term respiratory support. Though development of the double lumen cannula has played a significant role in reducing the ECMO footprint, all ECMO systems consist of a set of essential components. In order to overcome the resistance of the gas exchanger (oxygenator), a driving pressure in the form of either a centrifugal pump or a roller pump is needed. Large priming volumes and extensive circuit tubing necessitate the use of a heat exchanger to maintain normothermic blood temperature in the external environment, which altogether introduce an expansive foreign surface area. To mitigate the activation of biological pathways in the presence of artificial surfaces, the priming circuit is often supplemented with albumin, coating the internal surfaces with a protein layer, thereby reducing the inflammatory response. Still, a continuous anticoagulant drip (such as heparin), and frequent blood transfusions are necessary throughout the duration of support.

2.2. *Artificial Lungs*

Due to the limited availability of mechanical ventilation and ECMO therapies for patients with impaired respiratory function, these solutions for long-term pulmonary support are non-ideal. For routine bridge-to-transplant procedures, and cases of prolonged lung recovery, there are no suitable therapies; the sequelae from mechanical ventilation are undesirable, and ECMO can cause significant blood trauma. A more acceptable solution would be a means to provide adequate gas exchange, with low resistance, and excellent biocompatibility.

2.2.1. *Overview*

Each generation of gas exchange devices has brought about fundamental changes and insight for improving functional performance while diminishing adverse physiological consequences. Early designs effectively utilized the high surface area of percolating bubbles to oxygenate blood, but caused significant blood trauma in the form of mechanical destruction of red blood cells (RBCs) and platelets. Furthermore, these early bubble-oxygenators led to protein denaturation and coagulation disorders due to the direct contact of air with blood.¹¹⁻¹⁴

Other devices created a thin blood film (film-type oxygenators) where gas exchange occurred on the surface of the exposed blood film. But these designs required large surface areas, and consequently, high priming volumes. This configuration also resulted in complications arising from direct blood-air interactions.

The limitation of direct contact between air and the blood surface seen in previous generations of artificial lungs was resolved with the advent of membrane oxygenators. Similar to the native lung, where gas and blood phases are separated by the alveolar capillary wall, membrane oxygenators rely on a semi-permeable boundary by which oxygen and carbon dioxide species passively diffuse down concentration gradients. Modern microporous membrane oxygenators are designed to allow blood to flow around the outside of the hollow fibers while oxygen-rich sweep gas flows through the fiber lumen (see Figure 2.2).

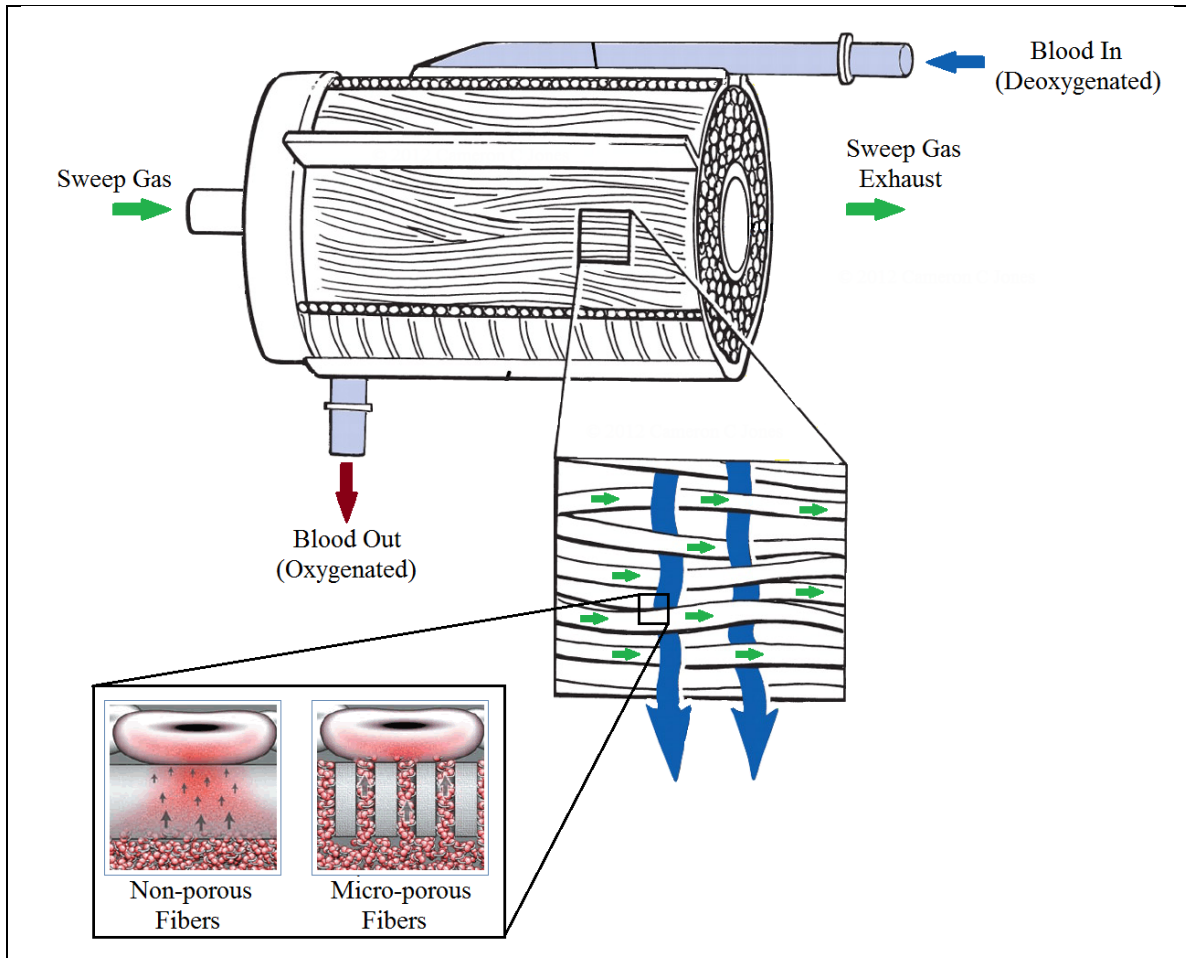


Figure 2.2: Dynamics of a Hollow-Fiber Membrane Oxygenator

Blood oxygenation occurs via simple diffusion of gaseous species as blood flows past an array of hollow-fibers. Most hollow-fiber oxygenators employ the strategy of blood flow across the outside surface, and sweep-gas through the fiber lumen. Non-porous fibers are resistant to plasma leakage but have lower gas exchange capacity per surface area than micro-porous fibers. Figure has been modified from Cohn.¹⁵

The fiber bundle is often comprised of thousands of hollow fibers (200–400 μ m OD) and provides highly effective gas exchange due to its microporous architecture and large surface area-to-priming volume ratio. The arrangement of fibers further improves diffusion due to the passive convective mixing of blood around the fibers.^{16,17} This efficiency translates into smaller devices, lower priming volumes, and a decreased blood-side resistance. Membrane oxygenators also allow for improved biocompatibility through surface coatings such as Teflon[®] and silicone, or surface-bound anticoagulants such as heparin and nitric oxide.^{8,18–20}

2.2.2. Principles of Gas Exchange

The lung's primary role is to provide oxygen (O_2) for physiological needs and to remove CO_2 , a metabolic waste product. Oxygen is poorly soluble in fluids, so most of the O_2 in the blood (about 98.5%) is transported bound to hemoglobin (Hb); the remainder is dissolved in the blood plasma. Carbon dioxide's solubility, likewise, is quite low in blood plasma (about 10%), but only about 30% is bound to Hb (as $HbCO_2$). The primary mode of transportation of CO_2 is in the form of bicarbonate ions (HCO_3^-), which constitutes about 60% of the CO_2 content in the venous circulation.

Gas exchange in the biological environment occurs entirely by passive diffusion according to partial pressure gradients, as characterized by Fick's first law of diffusion:

$$\dot{V} \propto \frac{A}{\Delta x} \mathcal{D} (P_1 - P_2), \quad (2.1)$$

where \dot{V} is the rate of gas transfer; A is the surface area; Δx is the membrane thickness; \mathcal{D} is the diffusivity; and P_1 and P_2 are the partial pressure differences across the semipermeable membrane. Gas diffusion in the native lungs occurs rapidly due to a large alveolar surface area (75 m^2 in adults) and thin alveolocapillary membrane ($\sim 0.5 \text{ }\mu\text{m}$).^{21,22} In fact, the principle diffusion barrier is not the membrane itself, but the diffusivity into blood plasma and nearby RBCs.

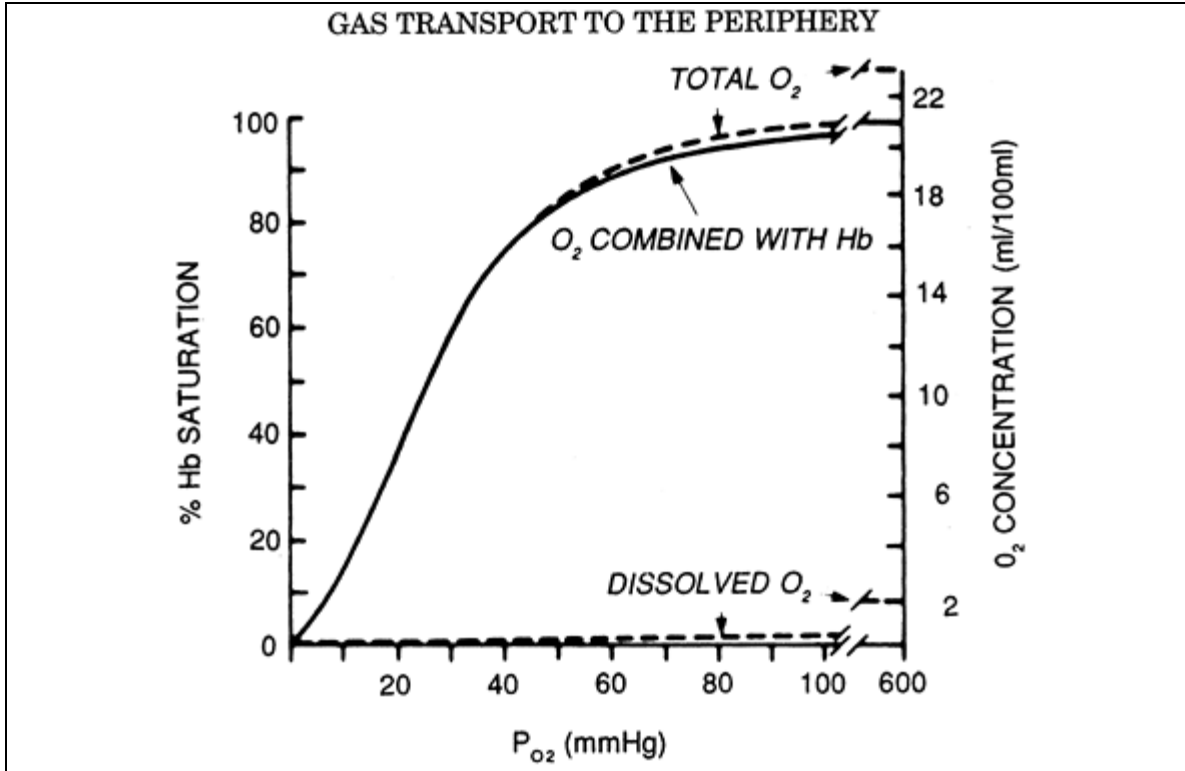


Figure 2.3: Hemoglobin Saturation Curve

The oxygen-hemoglobin saturation curve characterizes the bulk of the oxygen content carried in the blood. The primary driver for oxygen uptake and release is the O_2 partial pressure. This curve is representative of normal physiologic conditions (i.e., $pH = 7.4$; $P_{CO_2} = 40$ mmHg; and $37^\circ C$). Figure courtesy of West.²³

The primary driver for both gas dissolution and RBC uptake/release is the partial pressure gradient. For O_2 species, the oxyhemoglobin dissociation (or, saturation) curve characterizes the total O_2 content (bound as HbO_2 and dissolved) in an S-shaped relationship with the O_2 partial pressure, P_{O_2} , as shown in Figure 2.3. The oxyhemoglobin saturation (S_{O_2}) refers to the amount of O_2 bound to each of hemoglobin's four heme groups as a percentage value, with 100% being fully saturated. The amount of O_2 transferred in the blood is therefore the sum of both the oxygen bound to hemoglobin, and the amount dissolved in the blood plasma

$$O_2 \text{ content} = \lambda_{O_2} \cdot tHb \cdot \left(\frac{S_{O_2}}{100}\right) + \alpha_{O_2} \cdot (\Delta P_{O_2}), \quad (2.2)$$

where the hemoglobin binding capacity to oxygen $\lambda_{O_2} \equiv 1.34$; tHb is the Hb concentration; the oxygen solubility coefficient $\alpha_{O_2} \equiv 0.003$ at 37°C; where O_2 content is represented as mL O₂/dL blood.

In artificial lungs, the rate of O₂ transfer achieved by the gas exchange device can be quantified by multiplying the O₂ content from Eq. (2.2) by the volumetric blood flow rate through the device, Q_b , as

$$\dot{V}_{O_2} = 10 \cdot Q_b [1.34 \cdot tHb \cdot (\Delta S_{O_2}) + .003 \cdot (\Delta P_{O_2})], \quad (2.3)$$

where the multiplication factor of 10 serves as a unit conversion; and the change in S_{O_2} and P_{O_2} is observed as the device outlet–inlet values. Since CO₂ levels in the ambient air are at trace levels, a convenient method for quantifying the CO₂ removal performance of the artificial lung is to measure the partial pressure CO₂ in the exhaust gas (the sweep gas after passing through the membrane oxygenator) multiplied by the gas flow rate, Q_g , as

$$\dot{V}_{CO_2} = Q_g \left(\frac{EtCO_2}{P} \right), \quad (2.4)$$

where $EtCO_2$ is the end-tidal CO₂; P is the barometric pressure; and \dot{V}_{CO_2} is the rate of CO₂ transfer attained by the device.

2.2.3. Device Failure

The microporous membrane oxygenators used in today's CPB procedures have excellent gas transfer capabilities, require low priming volumes, and reduce the overall complexity of the extracorporeal circuit. However, these devices are susceptible to several complications which limit extended usage. Plasma leakage and thrombosis, for example, can affect the functional performance of the membrane oxygenator, resulting in decreased gas exchange efficiency and increased blood-side resistance.^{24–26}

During perfusion of the membrane oxygenator, phospholipids will adsorb on the hydrophobic fiber surface. While deposition of the phospholipid continues, surface tension across the submicron pores becomes compromised, allowing blood plasma to seep into the fiber lumen.^{25,26} As the severity of the plasma leakage progresses, the fiber lumens no longer serve as a conduit for sweep gas flow, resulting in diminished capacity for oxygenation and CO₂ removal. Some researchers are addressing this issue by investigating novel fiber-coating methods that would provide a thin barrier to prevent fiber wetting, while still utilizing the gas exchange efficiency of the microporous fibers.¹⁹ Others are focused on techniques for manufacturing very thin nonporous fibers with materials such as polymethylpentene (PMP) and silicone.^{19,20,26–28}

Additionally, thrombosis is a significant mechanism of device failure in membrane oxygenators, and typically develops later during device usage. Partially attributed to complications with biocompatibility and poor perfusion dynamics, thrombosis will increase device resistance and reduce the functional surface area for gas exchange.^{24,29–32} Like plasma leakage, thrombosis is a progressive development, but also carries the added concern for severe systemic complications resulting from potential thromboembolisms.

2.2.4. Discussion

The development of a suitable gas exchange device for long-term respiratory support can be summarized as an optimization process. While oxygenation capacity and CO₂ removal can be enhanced by increasing the effective surface area, greater foreign surface contact augments inflammatory and thrombogenic complications. Non-porous fiber membranes are resistant to plasma leakage, but result in lower gas exchange efficiency. Further, design configurations are continually striving to minimize sites of stagnancy and low flow regions, yet higher flow velocities can cause blood trauma.

As artificial lungs continue to develop and incorporate the collective experience from previous device generations, researchers are pressing on towards more successful outcomes with

extended usage. For instance, newer hollow fiber designs have significantly lower blood flow resistance than their flat sheet counterparts (adult hollow fiber membranes are typically 10–20 mmHg versus 100–150 mmHg for spiral silicone membrane oxygenators at clinically relevant flow rates)⁸, while possessing the additional benefit of passive convective mixing—permitting lower foreign surface area with comparable gas exchange capacity. Moreover, a simplified extracorporeal circuit is more amenable for the ultimate goal of ambulation. Patients that can achieve physical mobility benefit from improved pulmonary hygiene, increased immune response, better nutrition, and an overall increase in psychological outlook and quality of life. Whether used extracorporeally or implanted within the patient³³, various factors and design configurations must be evaluated in the development of a long-term device for pulmonary support.

2.3. *Computational Fluid Dynamics*

CFD is becoming a leading component in medical device design by providing predictions of device performance for a variety of research focuses. With increases in computational resources and numerical techniques, the accuracy and scope of CFD applications are rising. For membranous devices such as artificial lungs and hemodialyzers, CFD can provide cost-effective insight into device performance compared with conventional manufacturing and in vitro testing.³⁴

2.3.1. *Applications in Membrane Oxygenators*

Over the past couple of decades, numerical analytic assessments of the fluid flow using CFD have been utilized for evaluating characteristics of artificial lung designs such as pressure distribution, perfusion dynamics, and gas transport properties.^{29,30,34–42} With adjustments in simulated boundary and flow conditions, parameters influencing gas exchange efficiency and areas of recirculation are easily modified to obtain a device with desired properties.

Most CFD applications in membrane devices investigate steady-state conditions such as pressure distribution and velocity fields. Pathlines and streamlines yield information as to how blood flows through channels or around surfaces and variances in fluid velocities, particle residence time, and pressure contours have been used to assess global uniformity of the flow field.³⁶⁻⁴⁰

It is well known that non-uniform perfusion characteristics are undesirable, and it is often believed that flow path inhomogeneities can lead to stagnant blood flow and thrombosis in the artificial lung. Graefe et al. used CFD to improve the flow distribution by altering inlet and outlet port configurations³⁶, and Gartner et al. observed a correlation between thrombotic deposition in vivo and velocities predicted by CFD which resulted in low local convective mass transport.²⁹ Likewise, hollow-fiber hemodialyzers are designed to achieve uniform flow distribution in order to eliminate significant blood-to-dialysate flow mismatches, and to optimize diffusion efficiency.^{43,44}

2.3.2. Navier–Stokes Equations

The governing equations for viscous incompressible fluid flow are the momentum balance equation and conservation of mass, expressed respectively as

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \mu\Delta\mathbf{u} + F_B, \quad (2.5)$$

and

$$\nabla \cdot \mathbf{u} = 0, \quad (2.6)$$

where ρ is density; D/Dt is the substantial derivative; Δ is the Laplacian, and ∇ is the gradient operator in an appropriate coordinate system; $\mathbf{u} \equiv (u_1, u_2, u_3)^T$ is the velocity vector; p is pressure; μ is dynamic viscosity; and F_B is the body force term (e.g., gravitational forces and source terms). Though additional partial differential equations (PDEs) may be required to fully

describe the fluid flow (depending on assumptions held, such as energy conservation, etc.), Eqns. (2.5) and (2.6) will suffice for the problem statements in this dissertation.

The non-linear partial differential Navier–Stokes (N.–S.) equations are discretized in space and time, resulting in a set of algebraic equations which is solved iteratively at each discrete time point. For the incompressible N.–S. equations, most commercial CFD software, including ANSYS® Fluent, employ projection methods related to the SIMPLE method of Patankar⁴⁵, which initially solve the momentum equations in the absence of any pressure gradient terms, and then discretely invoke the divergence-free condition for computing pressure, thus implying that pressure can be calculated given the velocity field is known.⁴⁶ Yet, since pressure is not explicitly stated in the divergence-free equation, and only the gradient of pressure appears in the momentum equation, non-unique solutions may exist for given pressure distributions. Moreover, pressure computed in this way differs somewhat from physical pressure, especially at low values of Reynolds number.

2.3.3. *Grid Convergence*

Once changes in the flow properties following additional iterations are less than a prescribed value, the solution is considered converged. Residuals provide a measure of convergence of the conservation equations for the solution iterations; where typically three orders of magnitude in residual reduction are required for consideration of solution convergence.

Prior to concluding information from CFD simulations, it is important to verify that the solutions are mesh-independent. In steady-state (time-independent) simulations, refinement of the CFD grid with respect to spatial resolution conveys information regarding the discretization error of the simulation. As the grid spacing decreases (increasing grid resolution), spatial errors in the numerical results should converge to zero.

The test for grid convergence is based on Richardson’s extrapolation in which successively finer grid-spacing will eliminate dominant truncation errors.^{47,48} If $\{x_i\}_{i=1}^n$ represents

discrete grid points with a uniform step size $h = x_{i+1} - x_i$, and $f(x)$ is the exact (true) value of the function f (e.g., pressure loss) on a bounded domain $\Omega \subset \mathbb{R}$, then the corresponding numerical approximation $f^h(x)$ is evaluated as

$$f^h = f + \tau_1 h^{q_1} + \tau_2 h^{q_2} + \mathcal{O}(h^{q_3}), \quad (2.7)$$

where q_m is the order of the discretization method with $q_m \in \mathbb{R}$, $m = 1, 2, \dots, N$; and τ_m is the truncation error with $\tau_m \in \mathbb{C}$. The dominant error of f^h is therefore

$$e^h = f - f^h = -\tau_1 h^{q_1}. \quad (2.8)$$

In practice, q_1 is the theoretical order (or rate) of convergence of the numerical algorithm used in the CFD code, but grid sizes, boundary conditions, and numerical models can reduce the observed order of convergence. A direct evaluation of q_1 can be obtained from three numerical approximations as

$$q_1 = \frac{\ln\left(\frac{f^h - f^{h/r}}{f^{h/r} - f^{h/r^2}}\right)}{\ln(r)}, \quad (2.9)$$

where r is the refinement ratio, defined as

$$r = \frac{h_2}{h_1}, \quad (2.10)$$

in which h_1 is the finer (smaller) grid spacing (thus $r > 1$); and r is constant. Neglecting higher-order terms, the error of the grid with refinement r is therefore

$$e^{h/r} = f - f^{h/r} = -\tau_1 (h/r)^{q_1}. \quad (2.11)$$

Multiplying Eq. (2.8) by $(1/r)^{q_1}$ and subtracting from Eq. (2.11), the estimate of the continuum value becomes

$$f \cong f_1 + \frac{f_1 - f_2}{r^{q_1} - 1}, \quad (2.12)$$

where the numerical solution has computed the grid functions f_1 and f_2 for two grids of spacing h_1 and h_2 , respectively.

2.3.4. Discussion

Once grid convergence studies have been performed and errors from discretization methods have been addressed, CFD predictions may still deviate from physical results due to errors in geometric modeling and boundary conditions, as well as uncertainties in modeling constraints and simplifying assumptions. Moreover, since the governing equations are complex PDEs, multiple solutions may exist.

Generally speaking, CFD simulations do not serve as a replacement for experimental studies. Rather, CFD provides predictions of experimental variables which can help lower the amount of experimental testing and study costs; the accuracy of CFD simulations is ultimately limited by the strength of the underlying assumptions and the validity of the physical models incorporated into the governing equations.

When utilizing CFD for design analysis of membrane artificial lungs and other mass transfer devices, accurate predictions require validated flow and species transport properties. Often, experimental validation for these devices is based on comparison of numerical and experimental pressure distributions.^{30,40-42} For example, Funakubo et al. correlated overall numerical and experimental pressure drop³⁰, whereas both Gage et al. and Zhang et al. sampled pressure distribution from multiple sites drilled along the exterior housing of devices being tested.^{40,42} But neither of these studies acquired data validating the actual velocity field inside the fiber bundle despite its obvious importance.

Unfortunately, for membrane devices such as the artificial lung, direct observation of the perfusion dynamics is difficult to accomplish experimentally due to the opacity of the fiber bed, so limited emphasis has been placed on confirmation of numerically simulated velocity fields. While only a few studies have experimentally analyzed the perfusion dynamics in membranous devices, most of these data either lack spatial resolution or are restricted to unidirectional flow fields.^{43,44,49-52}

Since pressure is an integral property in incompressible N.-S. equation solutions, non-unique velocity fields might exist for a given pressure distribution. This principle, therefore, suggests that although it may be shown that experimental and numerical pressure fields are consistent, this does not necessarily confer accurate modeling of the numerical velocity field. It is the purpose of this dissertation to identify a method for obtaining experimental perfusion data through the fibrous medium in order to verify the accuracy of the numerical models.

2.4. *Flow through Porous Media*

A porous medium is a material that contains voids (or pores) in which the measure of void space is quantified by the porosity (ε), defined as

$$0 \leq \varepsilon = \frac{V_v}{V_v + V_s} = \frac{V_v}{V_t} \leq 1, \quad (2.13)$$

where V_v is the void volume; V_s is the solid volume fraction; and V_t is the total volume of the material. Often, V_v and V_s form two interpenetrating continua, but in applications of fluid transport, permeation through the porous medium is influenced by the effective porosity, ε_{eff} , which describes the void space that is accessible to flow. It is worth commenting, however, that although ε_{eff} may be significantly less than ε , it is not easily measured and often only addressed heuristically.

2.4.1. *Introduction*

The flow of fluids through porous media is frequently encountered in a variety of biomedical, chemical, and environmental applications, where understanding the dissipation of mechanical energy due to viscous and inertial resistance is extremely important. Physical characteristics of the fluid (viscosity and density), the porous medium (packing density, orientation, size, shape, and roughness), and experimental regime (rate of fluid flow, temperature, and pressure) are factors which influence permeation. Studies of fluid flow through fixed and

fluidized granular beds⁵³⁻⁵⁷ and fibrous media⁵⁸⁻⁶³ have provided empirical correlations which characterize pressure losses over a range of media porosities.

2.4.2. Functional Importance of the Reynolds Number

The Reynolds number (Re) is a dimensionless group which completely defines a system of fluid motion by providing a measure of inertial to viscous forces. Properties of the porous medium that influence flow behavior include porosity and the equivalent spherical diameter, d_p . Various flow regimes have been characterized for Newtonian fluids in porous media by the interstitial Reynolds number, Re_i ,^{55,56,64,65}

$$Re_i = \frac{\rho U d_p}{\mu(1 - \varepsilon)}, \quad (2.14)$$

where the following ranges are defined⁶⁵:

- (a) Darcy or *creeping* flow ($Re_i < 1$); linear relationship between pressure drop and flow rate; dominated by viscous forces. At $Re_i \cong 1$, boundary layers begin to form near the pore walls.
- (b) Inertial flow ($1 < Re_i < 10$); steady laminar flow; persists to $Re_i \cong 150$. Boundary layers become more pronounced and the formation of an inertial core contributes to a non-linear relationship between pressure drop and flow rate.
- (c) Unsteady flow ($150 < Re_i < 300$); laminar flow characterized by the formation of waves under certain porous arrangements.⁵⁶
- (d) Unsteady and chaotic flow ($Re_i \geq 300$); turbulent flow dominated by eddies.

The non-linear relationship between pressure drop and flow rate occurring at $Re_i \cong 5$ is attributed to pressure drag due to flow separation behind each particle (or fiber) and is not the onset of turbulence.⁶⁴ In fully turbulent flow ($Re_i \geq 300$), local losses due to flow expansions and contractions of the flow path are additive and dominate the overall pressure drop. For an

incompressible Newtonian fluid through a given porous medium, the Re_i is directly proportional to the superficial velocity, U , defined as

$$U = \frac{Q}{A}, \quad (2.15)$$

where Q is the flow rate; and A is the cross-sectional area of the fiber bundle.

2.4.3. Darcy's Law

According to Darcy's Law⁶⁶, the pressure drop per unit thickness, $\Delta p/\Delta x$, for slow flow through porous media, is dominated by viscous forces and can be described as a function of viscosity and permeability, k , given by

$$\frac{\Delta p}{\Delta x} = -\frac{\mu}{k}U. \quad (2.16)$$

The widely used Blake–Kozeny equation estimates pressure losses across a granular bed under Darcy flow as

$$\frac{\Delta p}{\Delta x} = -\left(\alpha \frac{(1-\varepsilon)^2 \mu U}{\varepsilon^3 d_p^2}\right), \quad (2.17)$$

where α is a constant; and $\varepsilon \leq 0.5$ for a regular arrangement of cylindrical fibers constituting the flow obstruction.⁶⁷

In recent work by Pacella et al., it was shown that for low flow rates, the Blake–Kozeny equation adequately predicted the Darcy permeability in layered fiber bundles (fibers stacked parallel or perpendicular to the flow field) for $\varepsilon = 0.47$ – 0.67 , but it is important to emphasize that a unique relation between k and ε is not usually available.⁶⁸

2.4.4. Ergun Equation

Under higher flow velocities (non-Darcy flow), kinetic energy losses occur due to direction changes in streamlines. The resistance created by inertial effects is proportional to the fluid velocity to the second power

$$\frac{\Delta p}{\Delta x} = - \left(\beta \frac{1 - \varepsilon \rho U^2}{\varepsilon^3 d_p} \right), \quad (2.18)$$

where β is a constant.⁶⁹ Total energy losses in porous beds can then be treated as a sum of viscous and kinetic losses:

$$\frac{\Delta p}{\Delta x} = - \left(a \frac{(1 - \varepsilon)^2 \mu U}{\varepsilon^3 d_p^2} + \beta \frac{1 - \varepsilon \rho U^2}{\varepsilon^3 d_p} \right), \quad (2.19)$$

which is often referred to as the Ergun equation.⁵⁴

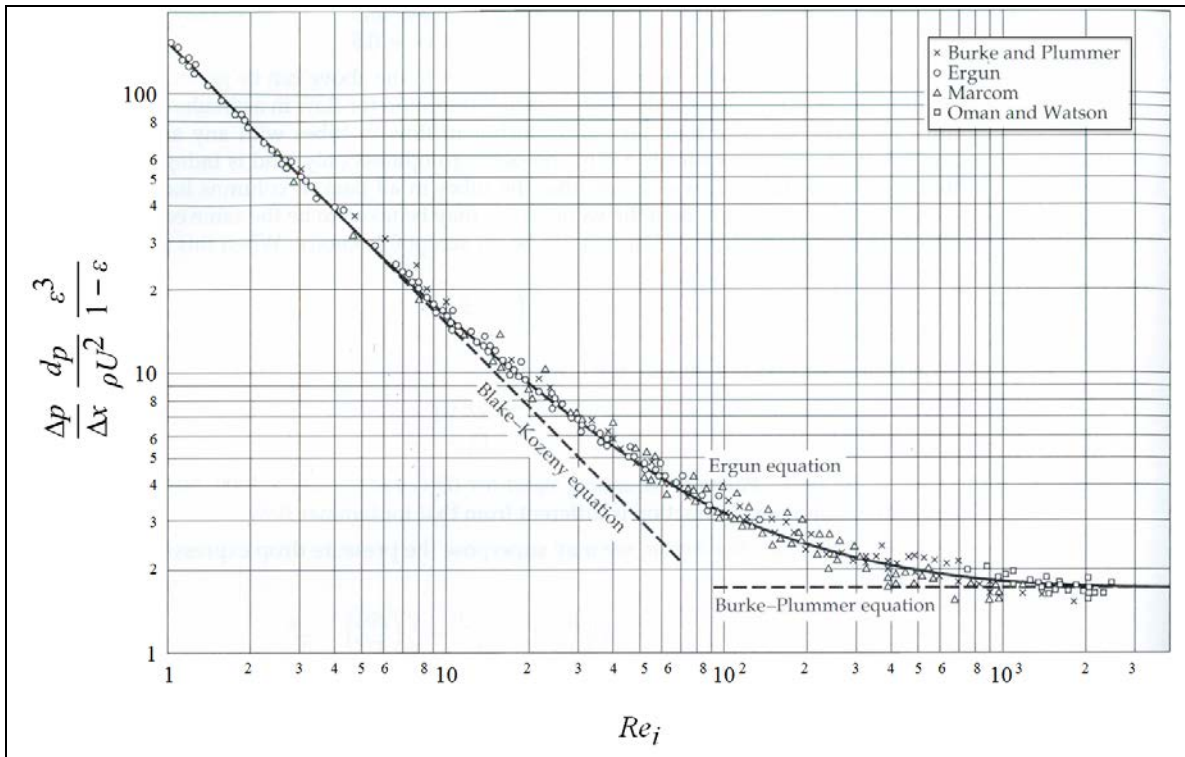


Figure 2.4: Pressure Drop through Porous Media according to the Reynolds Number

The pressure drop per unit thickness through porous media is characterized by three primary regimes: viscous-dominated (Blake–Kozeny); inertial-dominated (Burke–Plummer); and a transitional zone, $10 < Re_i < 10^3$, where both viscous and inertial forces contribute to pressure losses.

The dynamic relationship between viscous and inertial forces characterized by the Ergun equation is shown in Figure 2.4. For low Re_i , pressure losses are dominated by viscous forces (Blake–Kozeny), whereas inertial forces dictate momentum loss in higher flow regimes (Burke–Plummer); with noticeable derivation from strictly laminar flow occurring at $Re_i > \sim 10$.

2.4.5. Calculating Permeability and Inertial Coefficients

For a known cross-sectional area, the measured pressure drop for various flow rates can be fitted using a least-squares correlation of the form⁷⁰

$$\Delta p = aU^2 + bU, \quad (2.20)$$

where α and β of Eq. (2.19) can be evaluated as

$$\alpha = \left(\frac{\varepsilon^3}{(1-\varepsilon)^2} \frac{bd_p^2}{\mu\Delta x} \right), \quad (2.21)$$

and

$$\beta = \left(\frac{\varepsilon^3}{(1-\varepsilon)} \frac{ad_p}{\rho\Delta x} \right), \quad (2.22)$$

where again, Δx is taken to be the fiber bundle thickness; and U is the experimental superficial velocity. Studies by Ergun, Leva, and Macdonald et al., have empirically determined α to be in the range of 150–200, and β to be 1.75–4.0 for air flow in packed beds of spherical particles.^{54,55,71}

2.4.6. Numerical Approximation of Porous Media

In numerical analytical approximations of fluid flow through porous media, the associated pressure losses can be incorporated into the N.–S. momentum balance equation (Eq. 2.5) as a momentum sink, represented by a single source term, S :

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \mu\Delta\mathbf{u} + F_B + S, \quad (2.23)$$

where

$$S_i = -\left(\alpha \frac{(1-\varepsilon)^2}{\varepsilon^3 d_p^2} \mu u_i + \frac{1}{2} \beta \frac{(1-\varepsilon)}{\varepsilon^3 d_p} \rho U u_i \right). \quad (2.24)$$

The Ergun approximation is one of the most commonly used relationships for prescribing pressure drop through a fiber bundle according to simultaneous viscous and kinetic energy losses,

and studies have shown using S to characterize momentum loss in the fiber bundle are acceptable for predicting pressure distributions in hollow fiber membrane oxygenators.^{29,30,37–42}

For empirical studies of flow through fibrous media^{16,68} and those based on unit cell models^{72,73}, the equivalent spherical diameter is traditionally taken to be a function of ε and the hydraulic radius, r_h ,

$$d_p = 6r_h \frac{1 - \varepsilon}{\varepsilon}, \quad (2.25)$$

in which r_h is the is a ratio of V_v , and the wetted surface area, A_v ,

$$r_h = \frac{V_v}{A_v}. \quad (2.26)$$

Yet, for CFD studies of macroscopic flow through fibrous membranes, studies often define $d_p = d_o$, the fiber outer diameter; with generally good agreement to experimental pressure measurements.^{29,39–41}

2.4.7. Superficial, Physical, and Tortuous Velocity

Often the superficial velocity is used when defining permeability through porous media, because U does not require information regarding flow velocities inside the porous bed—since such information is usually unavailable. Yet, for purposes of validating the flow field inside a fibrous medium, such information is required.

In Blake’s derivation of the Darcy permeability (Eq. 2.17), he uses the Dupuit assumption that the velocity through the packed bed is inversely proportional to the bed porosity.^{67,74} This characterized by the average *physical* (or interstitial) velocity, \bar{U}_p , and can be calculated from U and ε by the formula

$$\bar{U}_p = \frac{U}{\varepsilon \cos(\theta)}, \quad (2.27)$$

where $\cos(\theta)$ represents the macroscopic flow path of the fluid, and is reflective of the packing arrangement of the bed and the effectiveness of the void space.⁵⁵ For a packed bed of spherical

particles, Carman showed that $\cos(\theta)$ was evenly distributed about the mean 45° angle⁵⁷; though, often $\cos(\theta)$ is taken to be equal to unity.^{54,64,70,75}

The actual microscopic flow path of the fluid around particles of the bed is the *tortuous* velocity, U_t . Inasmuch confusion exists around nomenclature (also referred to as pore-, and capillary-tube velocity) there also appears no unified verdict as to its value and meaning, but for discussion, the hydraulic tortuosity, τ , is defined as

$$\tau = \frac{L_e}{L} \geq 1, \quad (2.28)$$

where L_e is the mean effective length that the fluid particle travels; and L is the straight-line distance through the medium in the direction of the bulk flow (i.e., fiber bundle thickness).^{56,76}

Therefore,

$$U_t = \tau \bar{U}_p, \quad (2.29)$$

and it follows that

$$U_t = \tau \frac{U}{\varepsilon \cos(\theta)} \approx \tau \frac{U}{\varepsilon}. \quad (2.30)$$

Although tortuosity influences the permeability of the porous media, tortuosity itself is not an intrinsic property of the packed bed, but rather is contingent upon the method used in its calculation.⁷⁶⁻⁷⁸ For clarity, the differences between the various measurable velocities are illustrated in Figure 2.5.

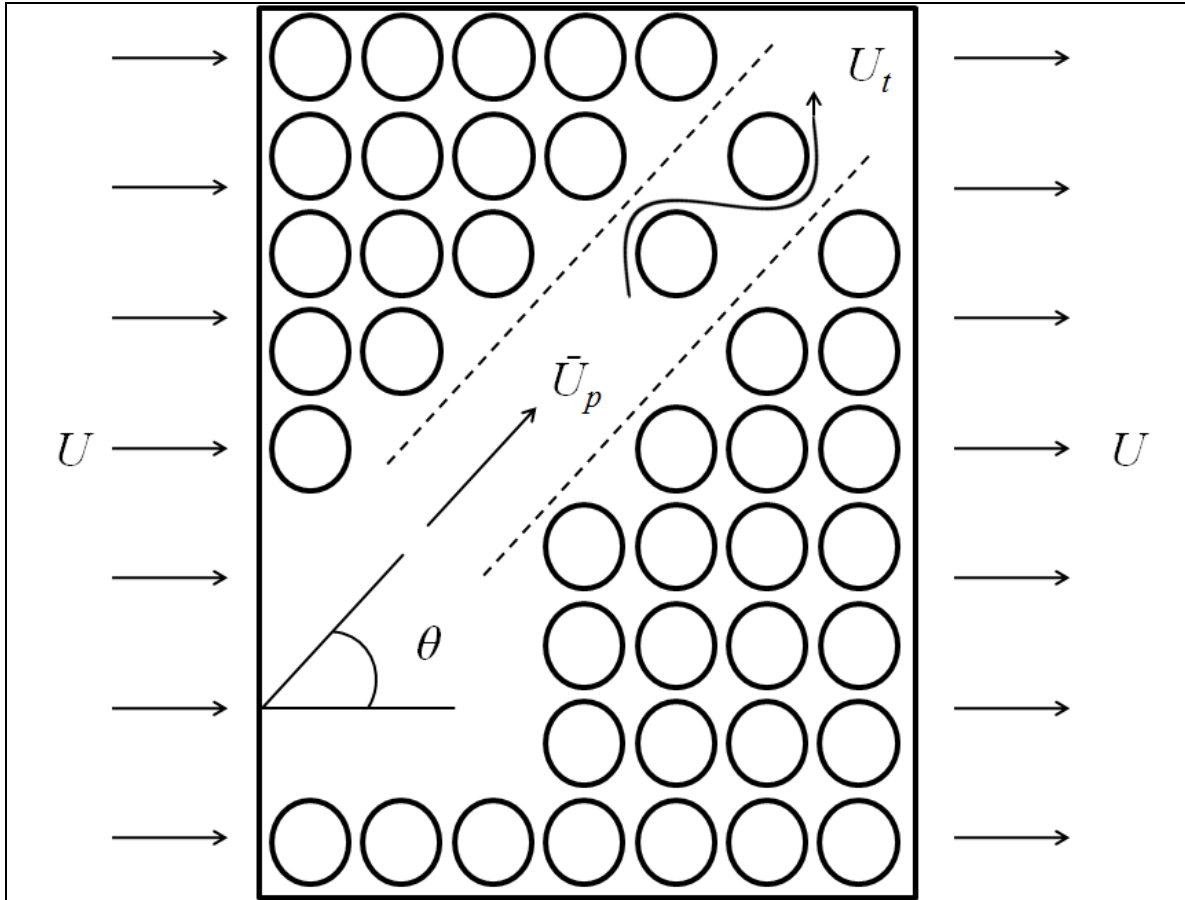


Figure 2.5: Superficial, Physical, and Tortuous Velocity

Superficial velocity, defined as the volumetric flow rate divided by the frontal surface area, is independent of porosity. Physical velocity accounts for the macroscopic flow through the porous media (a function of porosity) and may exist for $0 < \theta < 180^\circ$, assuming no backflow. Tortuous velocity details the microscopic flow permeating around individual porous elements. The relationship between the three measures of velocity is: $U \leq \bar{U}_p \leq U_t$.

2.4.8. Discussion

A porous medium can encompass a broad range of properties, composition, and environmental conditions; all of which are parameters that influence the permeability of a fluid (which itself may possess equal variety in nature). Because of the expansive variability in a system involving a permeating fluid, the permeability of a porous material is largely domain-specific. A brief survey of the literature will reveal numerous porous applications, configurations, and measurements of permeability in addition to those presented here.⁷⁹⁻⁸⁵

Nevertheless, permeability is principally governed by a viscous and inertial resistance to fluid flow, to the first- and second-order respectively. At low Re , permeability is linearly proportional to the pressure drop and superficial fluid velocity; whereas at higher rates, the viscous resistance becomes negligible and the inertial term dominates the pressure losses contributed by the fiber bundle.

Often, the porosity of the volume and the empirical coefficients in Eq. (2.19) are used to determine the permeability of the medium. Under this construct, porosity is an important parameter since it appears as second- and third-power terms. Though it is more difficult to ascertain in samples of unknown composition, porosity can easily be calculated from a device with prescribed manufacturing specifications. However, although fractional void volumes might be known, there may still exist uncertainties in the homogeneity, or isotropy of the packing arrangement, where $\varepsilon_{eff} \neq \varepsilon$. Therefore, for nontrivial cases, permeability is often verified using additional forms of measurement.

2.5. *Conventional Techniques for Visualizing Fluid Flow*

Since the performance of a gas exchanger depends heavily on the fluid flow properties through the fiber bundle, it is important to experimentally verify the permeability predictions of the device and fiber membrane. Typically, optical methods provide real-time, non-invasive measurements that are fairly simple in application, however, require specially designed housings or specific materials. Quasi-optical imaging modalities such as magnetic resonance imaging (MRI) and computed tomography (CT) can also provide relatively real-time acquisition and are less restrictive to membrane materials, but often demand more complex principles and/or reconstruction. Finally, several non-optical techniques offer good versatility, but usually necessitate invasive or destructive methods to both the device and/or flow field.

2.5.1. *Optical Methods*

Some of the most common optical techniques for visualizing fluid flow are based on particle tracking methods. These systems require the fluid—and its container—to be mostly transparent and the flow field to be seeded with neutrally buoyant tracer particles (usually a few microns in diameter). Depending on the particle density in the flow field, instantaneous velocities can be calculated either tracking individual particle displacements (particle tracking velocimetry, or PTV), or by performing statistical correlation methods (particle image velocimetry, or PIV). Modern PIV equipment are capable of operating at 1000s of frames per second (fps) and can provide highly resolved 3-D velocity fields by reconstructing projections from multiple viewing angles.⁸⁶⁻⁸⁹ Yet, for visibly opaque systems (as is the case for most porous media), optical methods such as PIV are unusable.

2.5.2. *Imaging Methods*

Most techniques for acquiring perfusion dynamics in membranous devices rely on various imaging modalities such as MRI or CT, since opacity is not a limitation with these instruments.^{43,44,49-51,90} Further, the methods for quantifying fluid flow using these imaging systems are generally available on most clinical and research units, which makes accessibility of these techniques quite useful for a range of applications.

In studies by Poh et al., MRI was used to investigate the effect of various configurations of flow baffles in a hollow fiber hemodialyzer and its effects on the fluid distribution for 0.2–1.0 L/min flow rates.⁴³ Similarly, Heese et al. used MRI to quantify fluid flow through both a hemodialyzer and an arterial blood filter at clinically relevant flow rates.⁵⁰ In both studies, the velocity-imaging techniques provided highly resolved insight into the spatial distribution of fluid flow, but were limited to mostly unidirectional flow. Further, due to long acquisition times, MRI measurements may be susceptible to motion artifacts.

Alternatively, Ronco et al. evaluated the uniformity and flow distribution in various hollow fiber hemodialyzers manufactured with different fiber configurations using a clinical CT scanner.⁴⁴ Uniformity was assessed based on 1-D regional velocity calculations at defined sections in a unidirectional flow field, and was useful for comparing velocities between flow in the center of the device and that moving parallel along the peripheral walls of the unit. Likewise, the study by Lu and Lu introduced an interesting method using CT imaging for tracking flow through a densely bundled hollow fiber membrane, but their technique employed various subjective thresholds and neglected to report additional analysis such as vector fields.⁵¹ Many other applications, advantages, and limitations of measuring fluid flow with techniques based on X-ray imaging will be discussed later in §2.6.

2.5.3. *Non-Optical Techniques*

Non-optical experimental techniques, such as those based on electrochemical and convective methods have been used to evaluate membrane perfusion. In a study assessing the uniformity of fluid flow for different membrane oxygenator designs, Hirano et al. placed 120 insulated copper electrodes throughout the fiber bundle and measured the time-dependent blood perfusion by varying the electrical potential of the infused fluid at 1 and 5 L/min.⁵² Alternatively, hot-wire anemometry (often, constant temperature anemometry, or CTA) has been used to monitor local velocity changes by measuring convective losses from a thin wire.⁴⁹ Both CTA and electrochemical means provide real-time flow information, but each would require wires or probes to be integrated into the membrane. In addition to imposing physical alterations which could influence the *normal* flow properties, these methods provide limited spatial resolution and only yield point-wise velocity information.

2.5.4. Discussion

Experimental evaluation of the perfusion dynamics through an opaque porous medium such as a membrane oxygenator is a nontrivial problem. Optical methods for quantifying fluid flow are ineffective in the absence of a transparent system, and non-optical methods generally require invasive actions, are limited to single-dimensional measurements, and have poor spatial resolution. Therefore, imaging modalities offer the most probable approach for obtaining quantitative information of flow through a porous medium.

Phase contrast (PC) MRI can generate 3-D flow fields with excellent spatial resolution, but require long acquisition times, which may be susceptible to fluctuations in the flow field. Yet, recently, advances in FLASH MRI have generated 2-D image sequences with temporal resolution of 20 ms and spatial resolution of 1.5–2 mm.⁹¹

Unfortunately, applications in MR imaging are limited by the physics of image acquisition and reconstruction. For example, localized velocity averaging occurs for pixels containing counter-current velocities or large discrepancies in nearby flow. Thus, in the case of the hollow-fiber membrane oxygenator, pixels that overlap the fluid zone and the void regions in neighboring fiber lumens would result in an underestimation of the local velocity. Further, the presence of magnetic materials will disrupt the image signal.

Alternatively, CT imaging uses less complex image reconstruction methods than MRI, requires lower acquisition times, and provides good spatial resolution. Despite the lower contrast discrimination and spatial resolution than MRI, the faster temporal response means greater invariance to motion artifacts, and for this reason, much has been done in developing methods for measuring fluid flow using X-ray techniques.

2.6. X-ray Imaging for Measuring Fluid Flow

X-ray imaging modalities (including CT, angiography, etc.) offer high spatio-temporal resolution which can be useful for real-time visualization of fluid flow. In medical applications,

rapid X-ray imaging techniques (up to 60 fps) are used for purposes of evaluating blood flow perfusion through a vascular network, particularly in the treatment of cerebral aneurysms.⁹²⁻¹⁰⁰

2.6.1. Principles of X-ray Imaging

X-rays are high energy photons of electromagnetic (EM) radiation that travel in straight lines unless scattered or absorbed by an obstructing material. An X-ray tube accelerates particles (electrons) toward the positive atomic nuclei of a target material (e.g., tungsten) whereby EM radiation is emitted from either deceleration of the electron or collision with an electron from the electron shell of the metal anode.¹⁰¹

In studies of fluid flow, contrast agents such as iodine, xenon, or barium are often used to enhance absorption of incident X-rays to improve image visibility via the photoelectric (PE) effect. Contrast agents appear dark in an X-ray image because their K-edge coincides with the peak energy in the X-ray spectral distribution.¹⁰¹ Numerous methods exist for detecting X-rays, but most rely on the PE absorption by a high molecular weight material (e.g., silver, phosphor, or selenium).¹⁰¹ Some modern detectors, like those used in radiography and fluoroscopy, use an X-ray scintillator and photocathode to convert incoming X-rays to visible light which can be digitally recorded by a video camera.¹⁰¹⁻¹⁰⁵ Other emerging technologies, likewise, deviate from the expensive, traditional film-type detectors with developments in solid state receptor plates, which convert the incoming EM radiation into a distribution of electric charges.^{101,106,107} It shall be noted, that though the principles and processing techniques contained herein can be applied to analysis of film-based X-ray images, the discussions that follow will assume the image to be digitized.

2.6.2. Digital Subtraction Angiography

A well-established X-ray technique for visualizing contrast-enhanced flow is digital subtraction angiography (DSA).^{100,108,109} In this approach, sequential radiographs of the contrast

perfusion are subtracted from a background (or “mask”) image taken prior to the infusion of the dye.^{110,111} Once background details are removed, the image histogram is digitally enhanced to optimize the image contrast visibility. This digital subtraction and enhancement technique improves the signal-to-noise ratio (SNR) by eliminating background structures that would otherwise convolute the flow features being imaged.

2.6.3. 2-D Videodensitometric Tracking Methods

In a series of X-ray images, the contrast density, $D(x, y, z, t)$, of each pixel (2-D “picture element”) is represented by a position in space at a given time. Since an X-ray image is a measure of the penetrating EM radiation, D is already integrated over the z -direction (this is discussed further in §2.6.4.). If the flow field can be assumed to be axisymmetric in the orthogonal x and y planes (such as flow in a blood vessel or pipe), the contrast density can be reduced to a single-dimension by integrating over the cross-section

$$D(x, t) = \int D(x, y, t) dy , \quad (2.31)$$

where $\{x, t\} \geq 0$.¹⁰⁰

In general, bolus tracking algorithms for 2-D densitometric images fall in two categories: time-density and distance-density methods. The former approach measures the transit time of the contrast medium from one [defined] region to another; the latter calculates the distance traveled between sequential frames (see Figure 2.6).^{92,94,100,109} Both time-density and distance-density methods assume the contrast is evenly and homogeneously distributed throughout the flow field; an assumption that is usually satisfied if the contrast medium has a viscosity approximately equal to the bulk fluid and has been infused at a rate greater-than or equal to the superficial velocity.⁹⁴

Time-density tracking methods rely on observing changes in image intensity as a function of time between two fixed regions of interest (ROIs). Numerous methods exist for measuring bolus arrival times, including time-of: peak-to-peak; threshold density; center of gravity; leading

edge detection; etc.¹⁰⁰ These algorithms have the advantage of being simple in construct, but are ill-suited for analysis of instantaneous velocities in pulsatile flows.¹⁰⁰

Distance-density methods, on the other hand, assess flow velocity by measuring the distance a certain intensity threshold (e.g., 50% of maximum) travels between two successive images. Distance-density methods are usually more accurate than time-density approaches since spatial resolution is typically higher than temporal resolution.⁹² Yet, analysis based on a defined threshold may be vulnerable to over-/under-estimation of the flow velocity, depending on how changes in contrast intensity are defined.¹⁰⁰

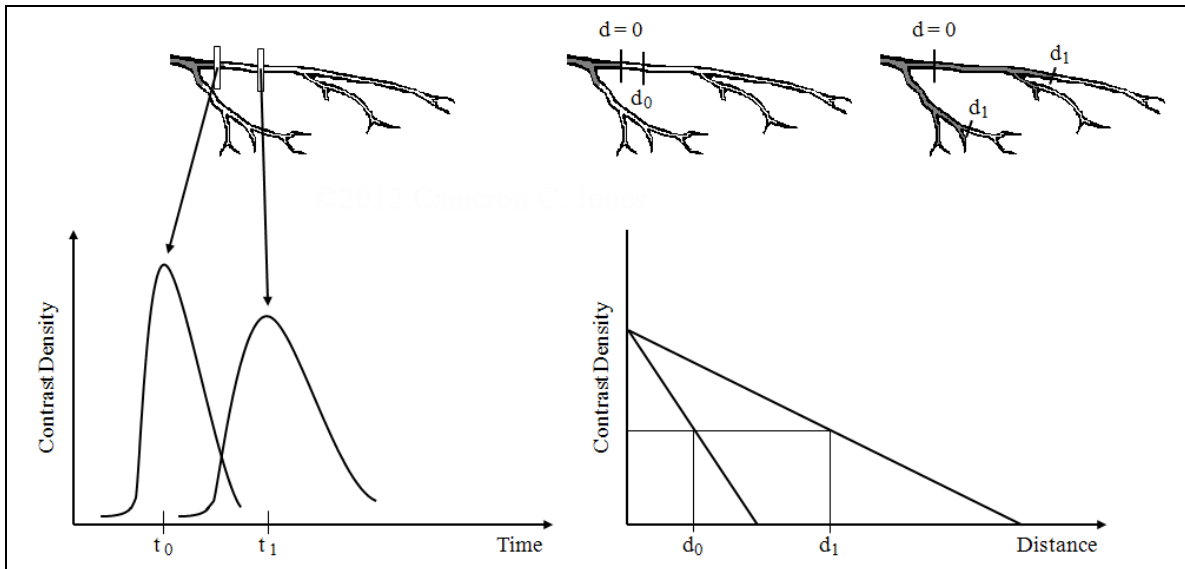


Figure 2.6: Time-Density and Distance-Density Bolus-Tracking Analysis Curves
 Time-density algorithms (left) measure the time difference between contrast-density curves for two defined ROIs; velocity is defined as the distance between the ROIs divided by Δt . Distance-density methods (right) calculate distance traveled between successive image frames according to a prescribed contrast threshold; velocity is defined as the distance between relative contrast levels divided by the time difference between successive images. The shape of the contrast density curves varies between inlet and outlet phases due to convective dispersion and/or branching of the infusion bolus.

2.6.4. Combining Multiple Imaging Projections

For a 3-D body, planar X-ray images provide little depth information since the X-ray projection is a sum of the total attenuation effects between the source and detector. Visual clues

such as surface texture, shading, and occlusion are absent from image projections, and depth perception is primarily achieved only from motion parallax in an image series.^{112,113} However, the combination of two-or-more projections from different vantage points can be useful in interpreting “z-plane” details through various reconstruction approaches.

The most common method of X-ray reconstruction—and the approach used by most CT scanners—is performed through Fourier back-projection or similar algebraic techniques.¹⁰¹ Using multiple projections spaced rotationally around an object, images are overlaid, or reconstructed, into 3-D renderings, with better clarity achieved from a greater number of image projections (see Figure 2.7).

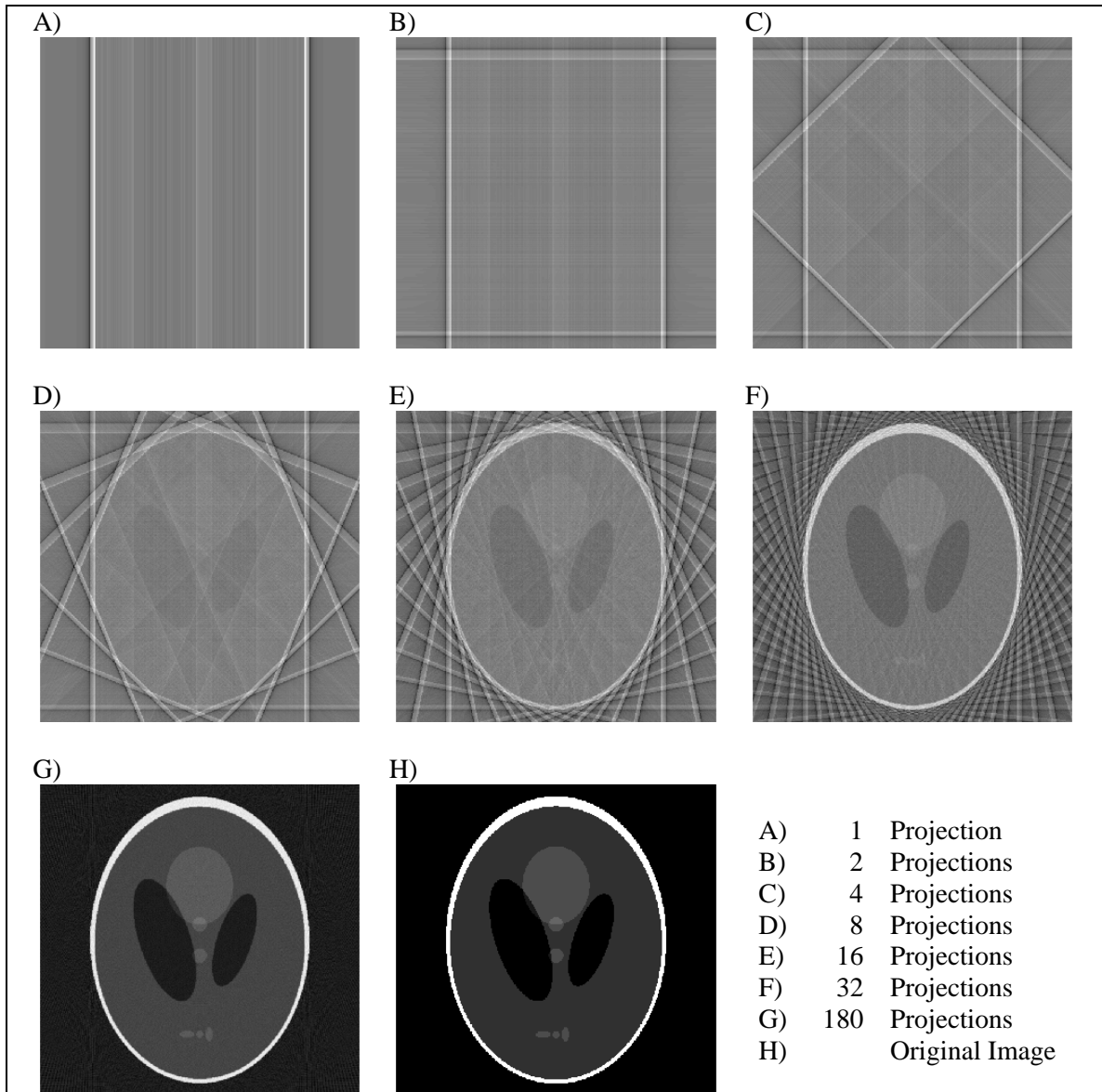


Figure 2.7: Slice Reconstruction through Filtered Back-Projection

The sequence of images from A to G illustrates the filtered back-projection method for the Shepp-Logan phantom using rotationally-distributed projection angles. Features of the image pattern become visible with higher number of reconstructed projections. The original phantom image is shown in frame H.

Tomographic techniques using filtered back-projection methods provide high spatial resolution, but only modest temporal response. Alternatively, details in the through-axis-plane may also be accomplished by stereoscopic methods without requiring tomographic reconstruction, whereby principles from visual depth perception are applied in the position of

two-or-more X-ray sources with angles of separation ranging from a few degrees to fully orthogonal.^{103,112-114} The offset between the two sources leads to slightly different images (viz., line integrals), and together generate a stereoscopic field of view (FOV), where corresponding markers in the stereoscopic image pair can be extrapolated to identify their 3-D position (see Figure 2.8).

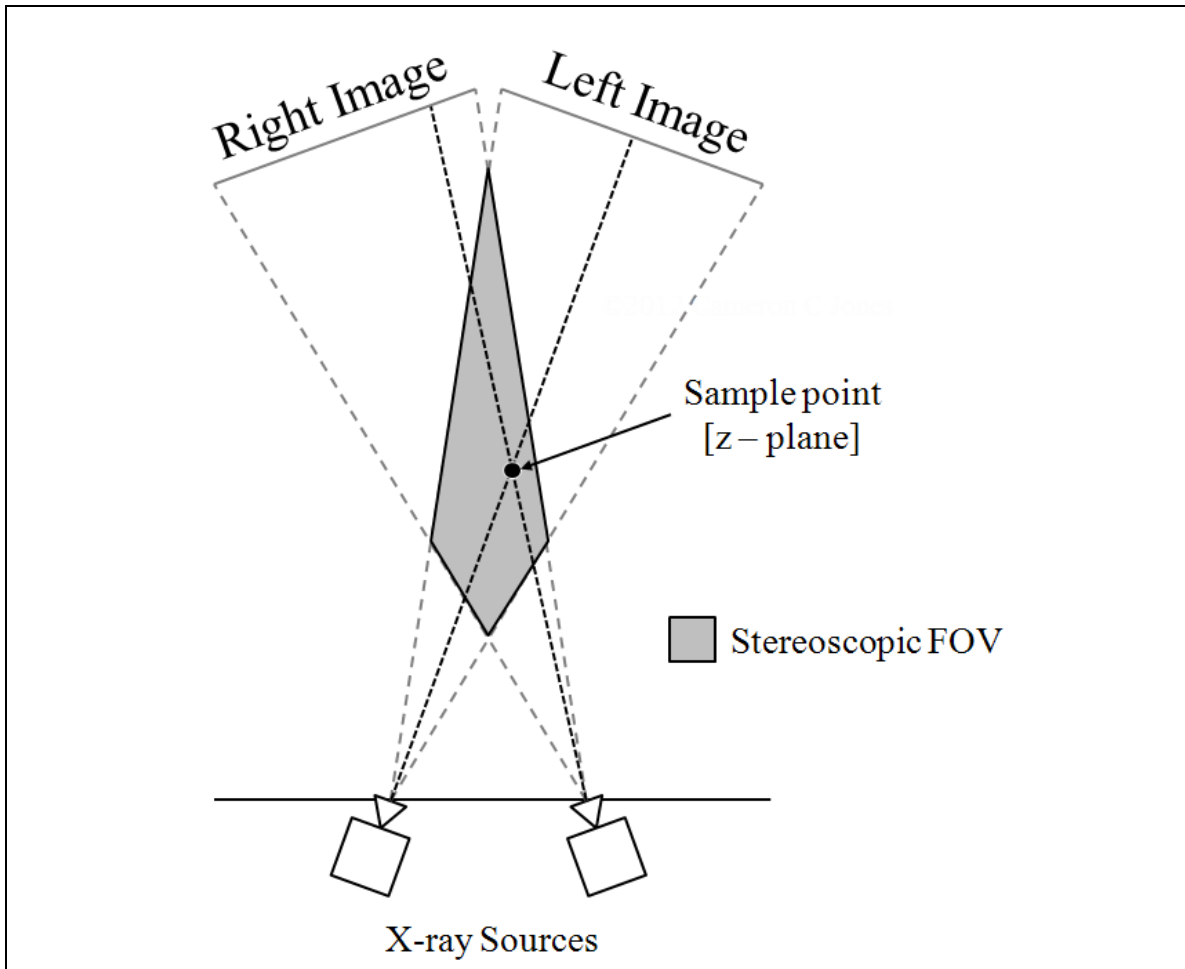


Figure 2.8: Illustration of Stereoscopic Reconstruction

Offsetting two-or-more X-ray sources/detectors about a central rotational axis provides alternative viewing angles useful for 3-D depth information. The area of overlap characterizes the stereoscopic FOV. Points existing in the FOV generate unique positions according to the combined locations between different image projections, and therefore allow spatial reconstruction in the stereoscopic z-plane. Depth resolution is dictated by the pixel spacing of image projections.

When unique features or markers in the stereoscopic projections are limited, or possibly indistinguishable, various strategies have been developed—mostly based on iterative processes. Since reconstruction from two planar images is an ill-formed problem, these techniques usually require a priori information and/or restrictive assumptions. For example, when reconstructing patient vasculature from biplane X-ray radiographs, Pellot et al. originally prescribed an elliptic approximation for the vessels, and then used an optimization algorithm to recursively deformed the shape according to back-projection gray-level (i.e., contrast density) values (Figure 2.9A).^{115,116} Similarly, Prause et al. used a 3-D anatomical model of the heart as the initial framework for iterative reconstruction of the left and right ventricles imaged with biplane angiography (Figure 2.9B).¹¹⁷

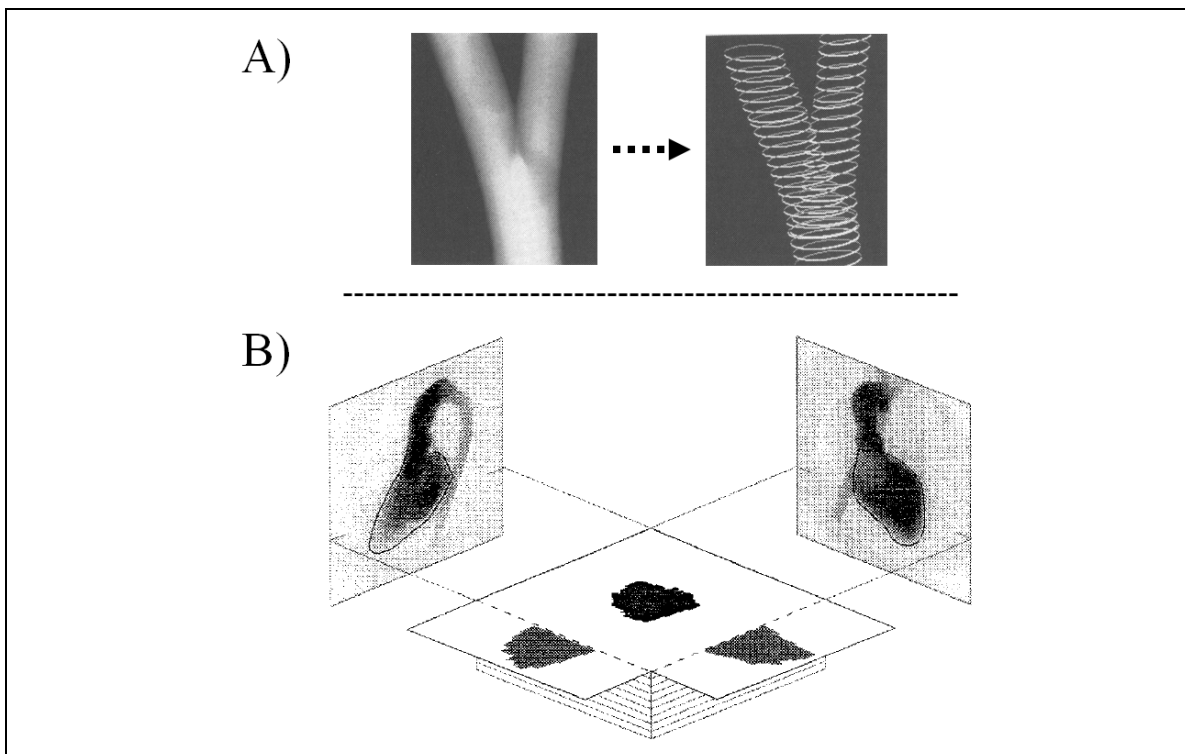


Figure 2.9: Iterative Reconstructive Methods from Biplane Angiography

A) Iterative reconstruction of a branching vessel based on deforming 2-D elliptical approximations. Image courtesy of Pellot et al.¹¹⁶ B) Binary reconstruction of the left ventricle from biplane image projections. The average shape of 20+ patient hearts served as an initial template by which further iterations were guided. Image courtesy of Prause et al.¹¹⁷ Most reconstruction methods construct volumetric models from stacks of 2-D data.

2.6.5. Discussion

The high spatio-temporal resolution of X-ray imaging techniques has encouraged numerous studies for measuring fluid flow in opaque systems that were previously inaccessible with optical methods. X-ray velocimetry utilizes the same principles as optical PTV and PIV and has been shown to be a reliable tool in both 2-D and 3-D applications.^{102–104,118} Further, techniques such as X-ray DSA improve visualization of the contrast-enhanced flow and can be performed real-time. Yet, selection of the appropriate X-ray imaging approach is study-specific. CT lacks temporal response for dynamic events and requires advanced reconstruction algorithms from multiple image projections, whereas non-tomographic means are generally simpler to implement, interpret, and faster in acquisition, but often require incorporation of approximations and assumptions. Features of the flow field such as pulsatility and the extent to which the flow is axisymmetrical will also narrow applicable measurement strategies.

For instance, bolus-tracking methods are, computationally, relatively simple, but are ill-suited for pulsatile flows and may result in incorrect measurements for non-plug flow, divergence in the contrast bolus, or in the case of backflow.¹⁰⁰ In addition to assuming a homogenous contrast distribution, most 3-D (and some 2-D) reconstruction methods require a linear relationship between changes in the X-ray intensity and density of the contrast agent.^{15,115,117,119}

Three-dimensional reconstruction of fluid flow using limited (e.g., two) projections can be useful in estimating the z-plane profile, but the problem remains ill-posed since two projections are not sufficient to resolve all of the through-plane information—even in sparse systems. Since X-ray projections represent sums of the total attenuation of the X-ray beam through the absorptive medium, images lack depth clues such as reflection, occlusion, and surface texture. Further, gray-level intensities, based on contrast density, might provide some depth perception, but are uninformative at steady maximum opacity of the image field due to unknown exact peak values.⁹² Often, strategies such as incorporating a priori information or reducing the system to a binary flow field can improve reconstruction approximations, but are limited to

application-specific venues.¹¹⁴⁻¹¹⁷ Still, others have developed algorithms based on motion parallax and convective dispersion models using rotational projections.^{99,120}

Although numerous assumptions and approximations are often utilized to resolve information from planar X-ray images, image projections do provide useful details regarding the dynamic behavior of fluid flow in opaque systems. In applications of CFD verification and validation, X-ray imaging techniques can be helpful for supporting numerical results, whereas the CFD simulations can be used to resolve specific features in the flow field.

2.7. *Conclusions*

The development of a suitable gas exchange device for long-term respiratory support requires evaluation of various factors and design configurations to improve the biocompatibility and performance of the device. Numerical simulations are valuable in providing predictions of physical data and can help lower the amount of experimental testing, but the accuracy of CFD studies is ultimately limited by the strength of the underlying models and assumptions incorporated into the governing equations of fluid flow. Validation of CFD results in membrane oxygenators traditionally has been based on comparison of numerical and experimental pressure distributions alone, since direct observation of the perfusion dynamics is difficult to accomplish due to the opacity of the fiber bed. However, the availability of X-ray imaging modalities to provide real-time, non-invasive assessments of fluid flow in opaque systems presents an opportunity for validating numerical simulations of fluid flow in membrane oxygenators.

3. CFD MODEL

3.1. Overview of the Test Device

A schematic of the general flow direction in the Affinity Oxygenator (Affinity NT[®] Oxygenator, Medtronic, Minneapolis, MN) is shown in Figure 3.1. Deoxygenated blood from the patient enters the device and passes through a heat exchanger to maintain body temperatures during device usage. The venous blood flows up the inlet tube, reverses direction in the inner flow channels, and permeates through the fiber bundle in a radial fashion. Gas exchange occurs in the fiber bundle region of the device due to local diffusion gradients, and is collected in an outer gap before exiting the outlet port.

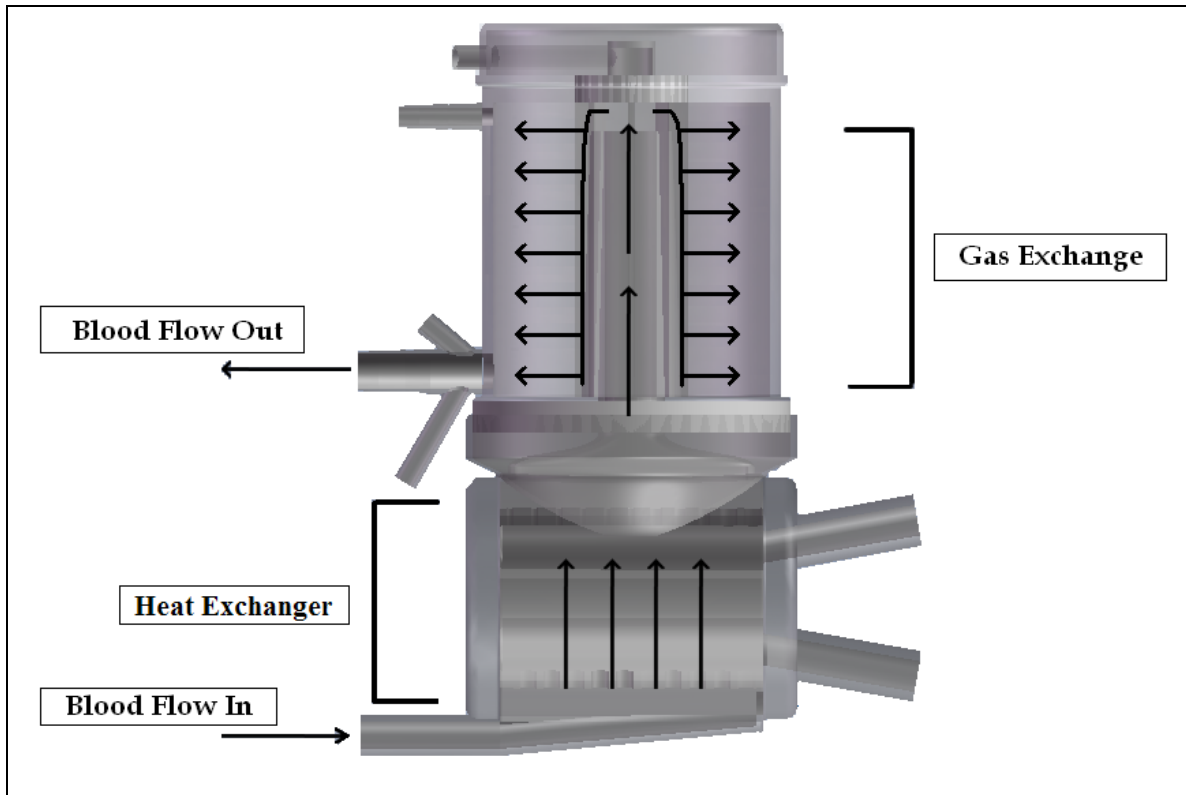


Figure 3.1: Representation of the Test Device

The general flow path in the Affinity NT[®] Oxygenator: Deoxygenated blood enters from the bottom left of the device, passes over the heat exchanger, and continues upward through the inlet tube. After reaching the top of the device, blood changes direction and flows down tapering channels, which facilitate an even distribution across the inner diameter of the fiber bundle. Gas exchange occurs in the fiber bundle and blood is collected in an outer gap, upon which the oxygenated blood is returned to the patient.

3.2. Computational Domain

Geometrical representations of the membrane oxygenator were drawn in a commercial CAD program (SolidWorks 2007, SolidWorks Corp., Concord, MA), and meshed (ANSYS[®] Meshing 12.1, ANSYS Inc., Canonsburg, PA) prior to solving Eqs. (2.6) and (2.23) for the flow field. Commercial CFD software (ANSYS[®] Fluent v12.1, ANSYS Inc.) was used to approximate the equations of fluid motion in the absence of any turbulence model. The geometric model consisted of three fluid regions: the inner flow (inlet tube and inner flow channels), an annular

fiber bundle, and the outer flow (an outer gap between the fiber bundle and housing and the outlet port); the heater was not included in the computational domain (see Figure 3.2).

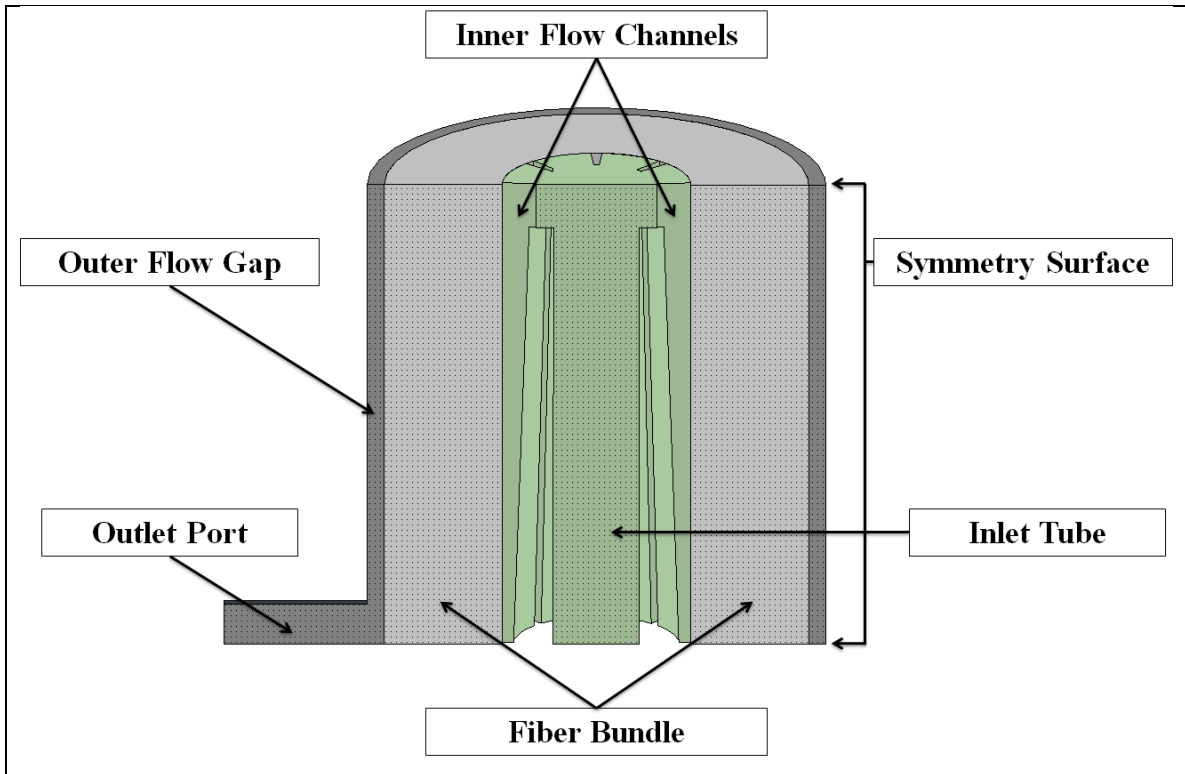


Figure 3.2: Computational Fluid Zones

The figure illustrates the three fluid zones defined in the CFD model: inlet tube and inner flow channels; fiber bundle; outer flow gap and outlet port. Inlet and outlet faces were defined normal to the inlet tube and outlet port, respectively. The dotted surface represents the plane of symmetry, modeled using a slip condition.

3.3. *Boundary Conditions and Model Assumptions*

CFD simulations were conducted for flow rates in the range 1.0–4.5 L/min. A constant inlet velocity (in m/s) was prescribed by dividing the volumetric flow rate by the inlet cross-sectional area (defined normal to the boundary), and a zeroed reference pressure at the outlet. To reduce computational expense, a symmetry (slip) condition was assigned to the problem domain boundaries formed from bisecting the oxygenator along the device’s symmetric midline as implied by Figure 3.2, and a no-slip condition was assigned to all solid surfaces (Figure 3.2).

Further, gravitational effects were not included in the computational model; hence $F_B \equiv 0$ in Eq. (2.23). Simulations were performed assuming standard operating pressure (101325 Pa).

Blood was modeled as a Newtonian fluid with constant viscosity (3.3 cP) and density (1050 kg/m³) undergoing incompressible laminar flow. The fiber membrane was treated as a lumped porous medium incorporated into the N.–S. equations as a single momentum sink according to the Ergun equation (2.24); the viscous and inertial permeability coefficients were defined using $d_p = d_o$. Due to the high packing density of the woven fiber bundle (45%), the permeability was assumed to be isotropic.^{39,40,42}

Fluent was executed using second-order spatial derivation approximations except for advective terms appearing in Eq. (2.23); these were treated with the usual first-order upwind procedure. A convergence criterion of 10^{-4} for the continuity equation and velocity component recoupling iterations was employed in achieving a steady flow solution. A summary of the boundary conditions and model parameters are listed in Table 3.1.

Table 3.1: CFD Boundary Conditions and Model Parameters

Inlet Velocity	0.098–0.441 m/s
Outlet Pressure	0 Pa
Symmetrical Midline	slip condition
All Solid Surfaces	$u_1 = u_2 = u_3 = 0$ (no slip condition)
Density, ρ	1050 kg/m ³
Viscosity, μ	0.0033 kg/m s (3.3 cP)
Outer Fiber Diameter, d_o	0.0003 m
Porosity, ε	0.45
Permeability, k	1.8074E–10 m ²
Darcy Coefficient, α	150
Inertial Loss Coefficient, β	3.5

Note: Inlet velocities are defined as the volumetric flow rate divided by the inlet cross-sectional area; the outlet pressure is relative to the inlet. Darcy and inertial loss coefficients are the conventional Ergun terms.⁵⁴

3.4. Grid Function Convergence

The grids used in this investigation were generated using unstructured 3-D tetrahedral/hybrid grid cells (see Figure 3.3). Convergence tests of the numerical solutions were performed with six grid sizes ranging from 20K to 1.72M cells, using the grid with the highest resolution as the theoretical *exact* solution, denoted \mathbf{f} (cf. Eq. 2.8). The error between \mathbf{f} and grid functions from coarser grids, \mathbf{f}^* , was quantified using a Euclidean norm composed of the dimensionless pressure drop, p^* , and maximum velocity, u^* , across the device; this norm was defined as

$$\|\mathbf{e}\|_2 = \left(\frac{1}{n} \sum_{i=1}^n (\mathbf{f}_i - \mathbf{f}_i^*)^2 \right)^{\frac{1}{2}}, \quad (3.1)$$

where $\mathbf{f}^* \equiv (p^*, u^*)^T$; and n is number of grid cells. Pressure and velocity components were made dimensionless by dividing by the dynamic pressure and inlet superficial velocity magnitude, U_{inlet} , respectively as

$$p^* = \frac{\Delta p}{\frac{1}{2} \rho U_{inlet}^2}, \quad (3.2)$$

and

$$u^* = \frac{u_{max}}{U_{inlet}}, \quad (3.3)$$

where u_{max} is the maximum velocity within the device.

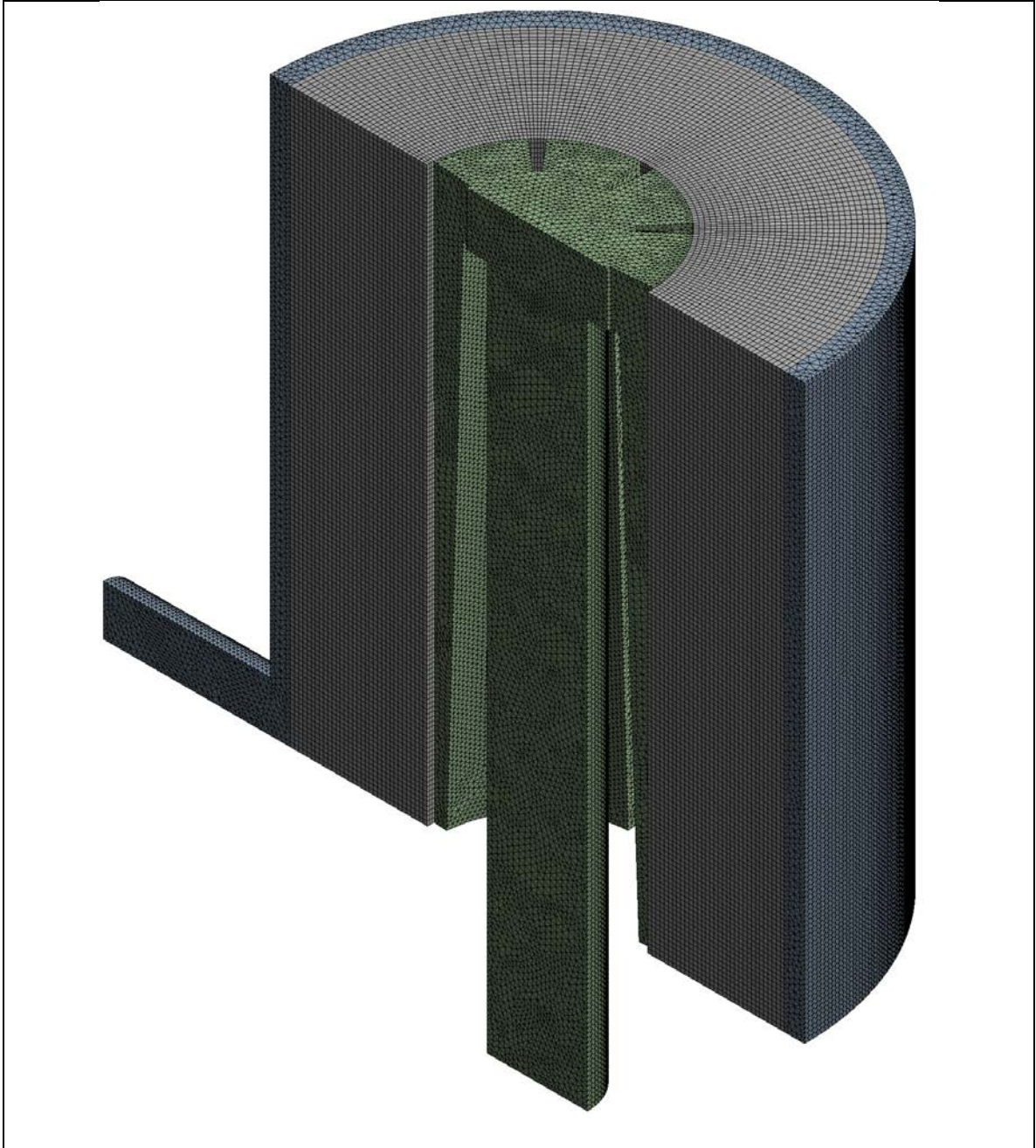


Figure 3.3: CFD Mesh

The conformal mesh contains 1.72M elements and employs a symmetry boundary condition along the device's vertical midline. The grid color identifies the three fluid zones: inlet (green), fibers (gray), and outlet (blue).

The error norms were fitted using a least-squares correlation, which characterized the convergence rate of the second-order numerical method as 1.69 (see Figure 3.4). The grid with the highest resolution was selected for all data representation, where pressure drop and velocity

vectors for the finest mesh are estimated to be within an error band of 3.3% and 1.6% of the continuum values, respectively. The numerical values used in the order of convergence calculations are listed in Table 3.2. Although stronger measures of convergence could be obtained from point-wise truncation error analysis, large disparities between velocities in the porous media region of the device versus the non-porous regions ($\sim 10^2$) would negate contributions of the velocity convergence in Eq. (3.1).

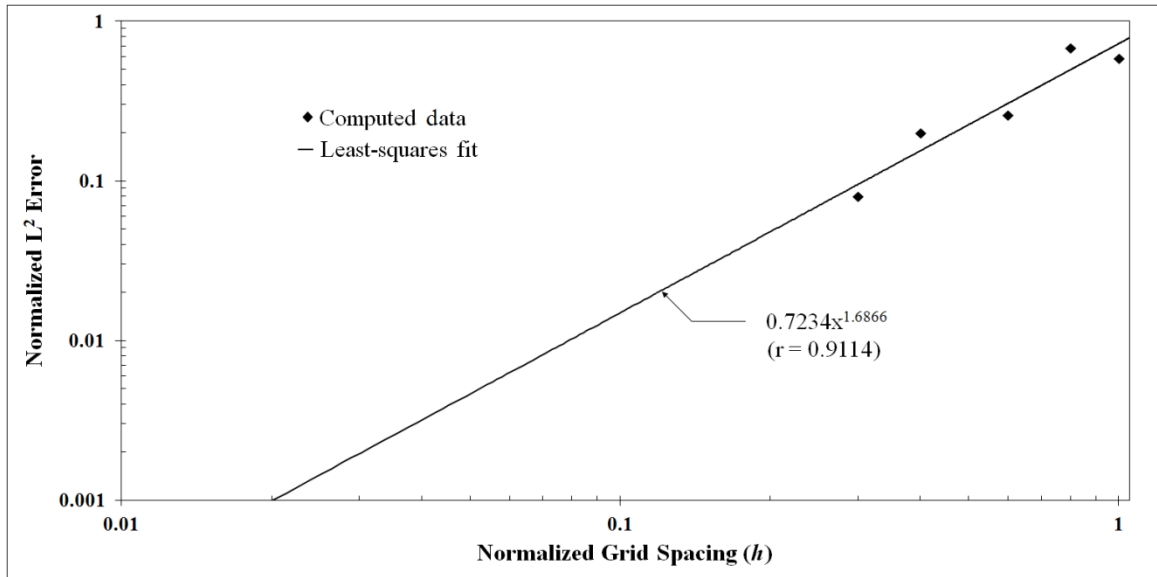


Figure 3.4: Grid Function Convergence

The order of convergence for the numerical studies is shown to be 1.69. Each of the coarser grid functions were subtracted from the finest mesh (theoretical solution) and their errors are plotted above. The pressure and velocity components of the mesh with the highest resolution were calculated to be within 3.3% and 1.6% of the continuum values, respectively.

Table 3.2: Grid Function Convergence Values

Elements ($\times 1000$)	Step Size, h	Δp (kg/m 2)	u_{max} (m/s)	p^*	u^*	error norm
27	1.0	1649.5	0.967	3.363	4.933	0.582
47	0.8	1670.3	0.954	3.494	4.866	0.679
90	0.6	1691.7	1.030	3.032	5.256	0.258
272	0.4	1749.9	1.050	3.022	5.355	0.198
597	0.3	1761.6	1.074	2.908	5.479	0.079
1720	0.2	1768.7	1.090	2.832	5.562	N/A

4. BIPLANE DSA

4.1. Experimental Setup

An important value to X-ray imaging analysis is that no modifications are required for the device in order to allow application of the technique (i.e., a “non-destructive” method). Since most constituents of commercial membrane artificial lungs are polymer-based materials, X-rays can easily penetrate the device, providing a peek at the internal flow properties. In biplane angiography, synchronous X-ray sources and detectors are positioned in orthogonal planes to provide real-time 2-D flow information. Further, the excellent spatio-temporal resolution of DSA is readily available in most clinical angiography suites. These features therefore enable insight into characteristics of the fluid flow through the membrane oxygenator that can be compared directly with CFD predictions.

4.1.1. Configuration of the X-ray System

A conventional clinical X-ray system was used for acquiring experimental perfusion data due to its availability, high temporal resolution, and relatively inexpensive application. Biplane DSA images were acquired by a Siemens Artis zee Biplane system (Siemens, Malvern, PA) at 7.5 fps for a contrast-enhanced flow through the Affinity membrane oxygenator. The oxygenator was oriented such that orthogonal X-ray acquisitions bisected the inlet flow channel in order to provide information regarding symmetry of the flow through the fiber membrane (see Figure 4.10).

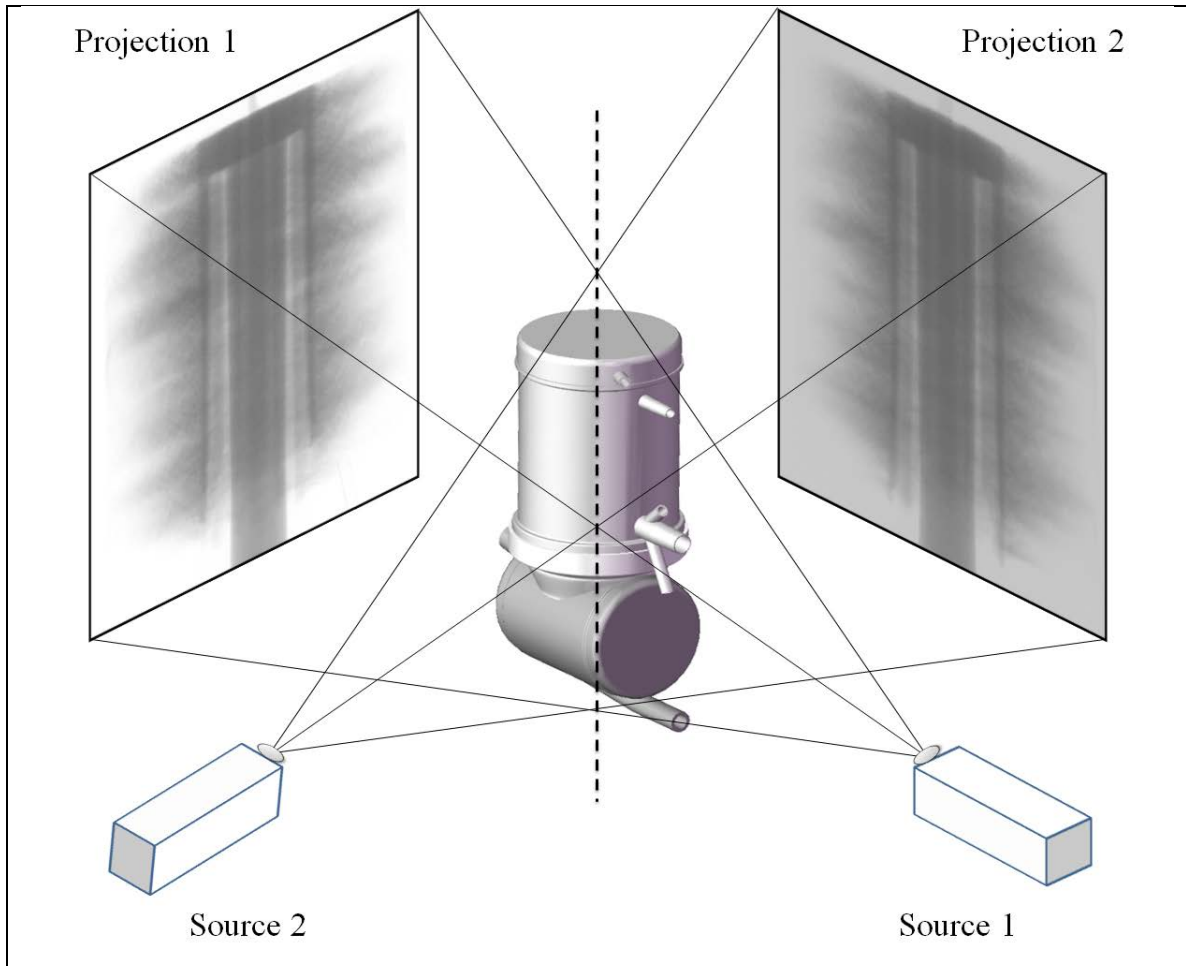


Figure 4.10: Experimental Setup of Biplane Projections

Orthogonal biplane X-ray sources were configured to obtain image projections along the inlet tube at 7.5 fps.

4.1.2. Configuration of the Fluid Circuit

The bench circuit consisted of a 37% by weight aqueous-glycerin solution at 21°C (comparable to normal patient blood viscosity) and was circulated with a centrifugal pump (CentriMag®, Levitronix®, Zurich, Switzerland) at flow rates ranging 1.0–4.5 L/min as measured by a Transonic Systems bypass flow meter (HT110, Transonic Systems Inc., Ithaca, NY). One hundred fifty milliliters of an iodinated contrast agent iohexol (Omnipaque™ 300 mgI/mL, GE Healthcare Inc., Waukesha, WI), diluted to a room temperature viscosity of 3.3 cP, was infused through an inline port 1 m upstream of the oxygenator inlet at the same flow rate as the bench

circuit. Biplane images were acquired for 20 s to allow full visualization of the dye entering and leaving the device. A summary of the study parameters are in Table 4.1.

Table 4.1: Experiment Study Parameters

Flow Rate, Q	1.0–4.5 L/min
Blood Analogue	37% w/v aqueous-glycerin solution
Ambient Temperature	21°C
Contrast Volume	150 mL

4.2. Analysis of Angiographic Images

All image analysis and post-processing was performed in MATLAB[®] (v.7.12 R2011a, The MathWorks, Inc., Natick, MA) using a custom code (pseudo-language algorithms are provided in Appendix C).

4.2.1. Image Processing

The study-generated DICOM image files were imported into MATLAB and the maximum background intensity (threshold) was subtracted from all subsequent images. All pixel values above the threshold were assigned a value of 1; those below, a value of 0. The binary images were then digitally rotated and cropped for proper spatial alignment with the complementary biplane projection using known markers on the oxygenator housing. The spatial and temporal resolution was known to be 0.2 mm and 0.133 s, respectively.

Prior to analyzing the flow field data, a median filter¹²¹ using the nearest 3×3 neighboring points was used to reduce noise and preserve edges of the digital images for improved handling of the velocity estimates.¹²² In 2-D form, the median filter consists of an $M \times M$ window (M must be odd), where the median of the discrete sequence of elements replaces the middle value of the window. Following spatial smoothing by the median filter, a flood-fill

operation was used to fill remaining “holes” in the contrast image. These functions are illustrated in Figure 4.2.

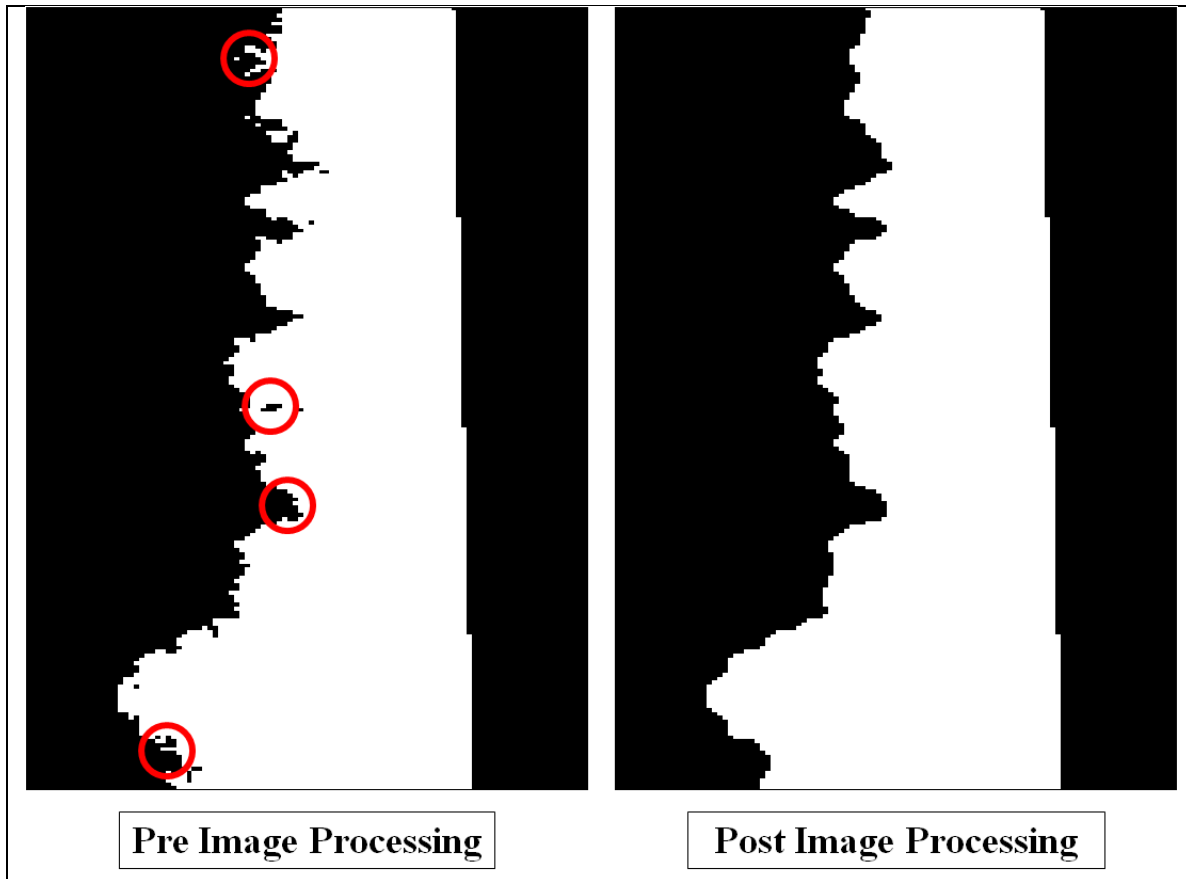


Figure 4.2: Image Processing Operations

Prior to flow-field measurements, the image dataset undergoes several iterations of image smoothing. Rough pixelations are smoothed with a median filter using a 3×3 pixel window, and a flood-fill operation is used to fill holes in the contrast medium. Representative regions have been highlighted for clarity.

4.2.2. *Maximum Cross Correlation Method*

Propagation of the contrast wave through the membrane was tracked using a maximum cross correlation (MCC) method, which identifies pattern-based correlations between two sequential time-dependent images. Though the MCC method is a non-traditional application of X-ray image analysis, the method has been shown to be a reliable tool for assessing progression of flow.^{123–125} In this approach, a grid is mapped over the image at a time step, t_i , dividing the

image projection into a series of templates. For each template, a larger grid (referred to as the search window) is mapped in the subsequent X-ray image, t_{i+1} ; whereby the corresponding search window in t_{i+1} is large enough to encompass the maximum distance the flow might progress between sequential image frames (see Figure 4.3). In a 2-D projection, each element in the resulting matrix contains a correlation value from -1.0 to 1.0 based on the degree of similarity between the image and the template features. The normalized correlation coefficient matrix, C , is obtained by

$$C(v, w) = \frac{\sum_{x,y} \{ [I(x, y) - \bar{I}_{v,w}] [T(x-v, y-w) - \bar{T}] \}}{\sqrt{\sum_{x,y} [I(x, y) - \bar{I}_{v,w}]^2 \sum_{x,y} [T(x-v, y-w) - \bar{T}]^2}}, \quad (4.1)$$

where the pixel values of the search window are $I(x, y)$; $\bar{I}_{v,w}$ is the average value of $I(x, y)$ in the region under the template, T , positioned at v, w ; and \bar{T} is the average template value.¹²⁶ The maximum value of the cross correlation matrix indicates the most probable likelihood of the template's displacement, while the vector at each grid point is given by

$$c = \frac{\sqrt{(v)^2 + (w)^2}}{\Delta t}, \quad (4.2)$$

and

$$\theta = \arctan\left(\frac{w}{v}\right), \quad (4.3)$$

where c is the velocity magnitude (in pixels/s); t is the time between sequential images (1/fps); and θ is the vector orientation.

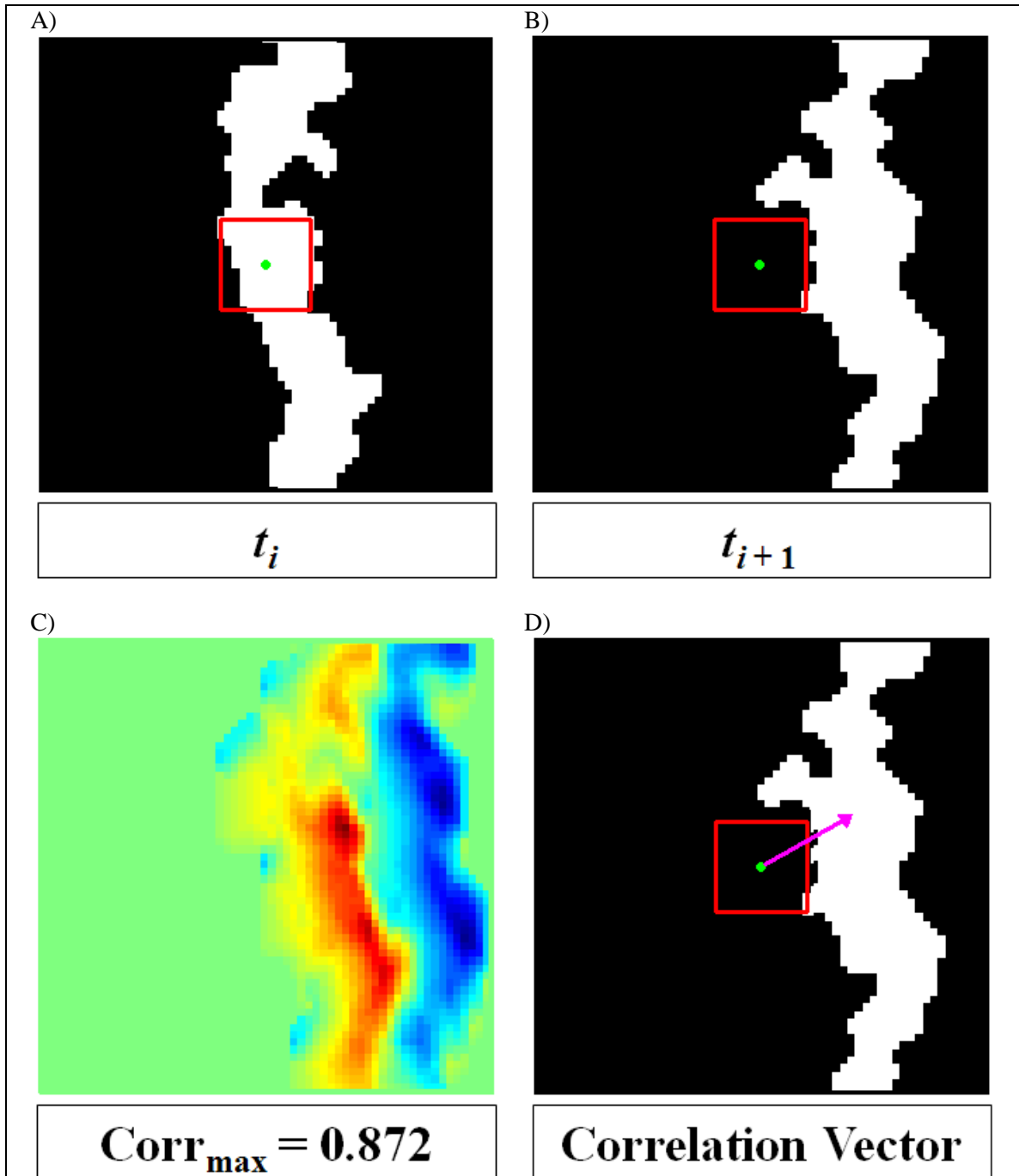


Figure 4.3: Maximum Cross Correlation Function

The MCC method identifies the most likely displacement between two successive ROIs according to the correlation of pattern-based geometries. A) The red box indicates the template at time t_i , centered at a specific grid point. B) The flow field is advancing from left-to-right at position t_{i+1} ; the template from the previous time step is shown for reference and the entire field-of-view characterizes the search window. C) A correlation map portrays the correlation of the template (red box of t_i) with the new flow field in t_{i+1} . D) The vector according to the maximum correlation is drawn for the given grid point.

4.2.3. MCC Filtering Algorithms

The MCC method requires the user to define both template and search window sizes, with the only two conditions being: the search window encompasses the maximum distance the template would travel in a given time-step; and the template contains enough features to be distinguishable. Therefore, it is no surprise that the values one chooses for template and search window boundaries can influence the final output. Moreover, it is entirely likely that false positive correlations may exist, whereby the template corresponds better with a random structure of the flow field than with the true fluid movement.

To reduce the subjectivity in defining template and search window sizes, iterations were run using rectangular template sizes ranging from 5 to 53 pixels per side (*pps*), and search window sizes ranging from 13 to 149 *pps* (recall, digital resolution is 5.333 pixels/mm). For each template size, the average velocity in the fiber bundle calculated by each search window was averaged and is graphed in Figure 4.4, where the error bars indicate standard deviation between the averages computed by the different search windows. As might be expected from a pattern-based correlation method, larger templates will be more robust against false positives. Indeed, as represented in Figure 4.4, increasing the template size led to more constant and consistent correlations. The maximum template (53 *pps*) and search window (149 *pps*) sizes were used for the analysis in this study.

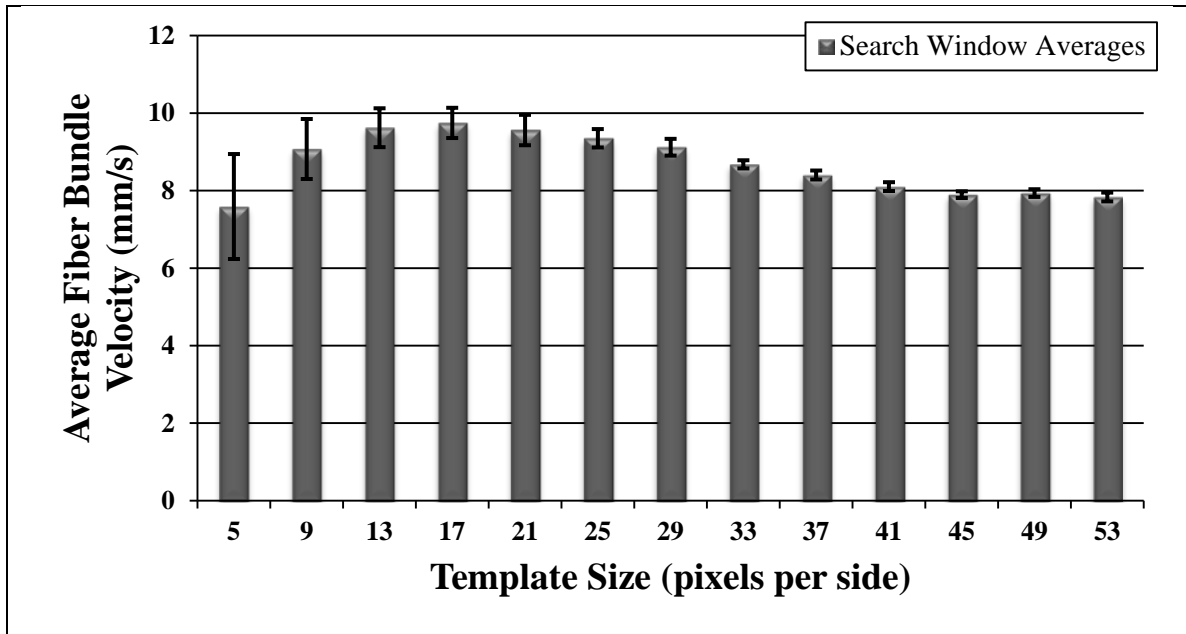


Figure 4.4: Convergence Testing of MCC Variables

Two-dimensional contrast perfusion through the fiber bundle was discretized into hundreds of nodes with rectangular template sizes ranging from 5 to 53 *pps* defined at each node. For a given template, different search windows ranging from 13 to 149 *pps* were used to define the boundaries of the correlation-based tracking algorithm. Average velocity magnitudes in the fiber bundle from each of the search windows were themselves averaged and have been graphed in the figure above. The error bars indicate standard deviation between the averages computed by the different search windows.

Since it is possible that a template pattern may produce a high correlation with a random pattern in the following image, spatially incoherent vectors were filtered using various algorithms. Resultant vectors in the inner flow channel, for example, clearly should not correlate with positions upstream of the flow field and were therefore removed. Similarly, vectors with magnitudes that differed by more than 2 standard deviations from surrounding vectors were declared inaccurate. Filtering rules such as these were run tangentially with the cross-correlation code such that if an erroneous vector was detected, the next highest correlation that was not in violation of one of the filters was accepted. Finally, only vectors with a cross-correlation greater than 75% were included. No interpolation scheme was employed; therefore, sparsities in the vector field do not indicate a region with no perfusion, but rather a nodal point for which no acceptable correlation was found.

4.2.4. *Point Cloud Reconstruction*

The orthogonal biplane projections provide only limited insight into the 3-D flow properties (cf. §2.6.4.), but with reasonable assumptions, useful qualitative information may be obtained. For each time step, the cumulative perfusion for both image projections was outlined by a 2-D boundary, which traced the outer edge of the fluid perfusion and the internal features of the oxygenator housing. The boundaries were divided into horizontal slices with a thickness of 1 pixel. Pixels for each of the biplane projections served as the nodal points in a cubic spline interpolation—existing in the horizontal, or z-direction (see Figure 4.5). The interpolant was constructed with a high resolution of points to help improve smoothing and reconstruction.

For each point in the interpolant, its 3-D spatial position and normal component were stored in an $N \times 6$ matrix, where N indicates the number of slices in a given time step. Since the dye is assumed to be homogeneously dispersed in the bulk fluid, and earlier imaging processing steps eliminated “holes” in the binary perfusion image, the maximum number of ‘ON’ pixels for each horizontal slice ranged from 1 to 6. The resulting point cloud was exported to MeshLab (v.1.3.0, Visual Computing Lab, ISTI-CNR) which overlaid surfaces on the 3-D point set.

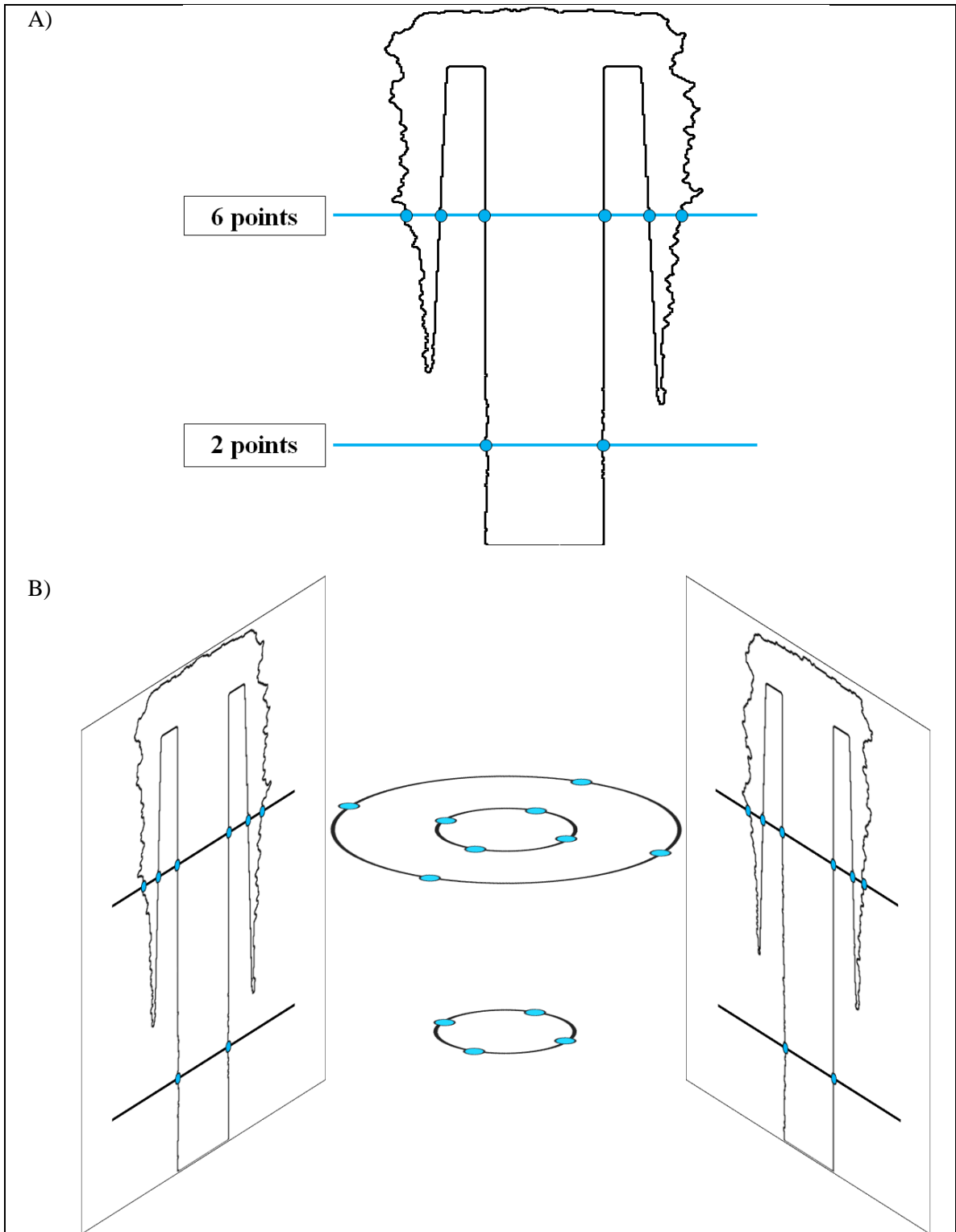


Figure 4.5: Biplane Image Interpolation Technique

A) The interpolation algorithm required exactly 2 or 6 matching points between the two biplane projections for reconstruction of the interpolant. B) Corresponding points from each horizontal row were used as vertices for the cubic spline interpolation existing in the z-plane.

4.2.5. *Image Processing Assumptions*

As with all methods of contrast-enhanced flow, the contrast agent has been assumed to be homogeneously dispersed throughout the transport fluid. By ensuring the dye has approximately the same viscosity as the fluid medium and infusion rates match the superficial bulk velocity, this condition is usually satisfied.

In both MCC and point-cloud reconstruction studies, the most significant requirement was to assume that the leading (or outer) edge of the dye was in the projected image plane. Since densitometric images are integral projections, no information exists for through-plane velocities. However, based on the annular structure of the Affinity Oxygenator and the woven fiber bundle, the shortest flow path, when observing the device's cross-section, would be direct lateral perfusion (i.e., radial perfusion). This was confirmed by comparing the orthogonal biplane projections, which showed nearly identical densitometric profiles (discussed further in §5.2.3.). Furthermore, since the assumption of visually 2-D flow provided no details of through-plane flow movement, the binary images were defined to be in the bisecting image plane.

5. EVALUATION OF EXPERIMENTAL DATA

5.1. *Experimental Pressure Drop*

Pressure was measured multiple times ($n = 3$) for each flow rate using a TruWave transducer kit (Edwards LifeSciences, Irvine, CA) connected to a full-bridge module (NI-9237, National Instruments, Austin, TX) on a CompactDAQ chassis (NI cDAQ-9172, National Instruments), and recorded using LabVIEW™ (v.8.6, National Instruments). The pressure drop across the heat exchanger constituted a significant portion of the total pressure drop of the device, as shown in Figure 5.1. The combined resistance from both components of the device (heat exchanger and fiber bundle) will affect parameters such as shear stresses and device-induced platelet activation, but only the pressure losses imparted by the fiber bundle determine the permeability through the fibrous medium. Since no sites existed between the integrated heat exchanger and fiber bundle, and separating the two units would likely cause unintended flow disturbances upstream of the fiber bundle. Therefore, the pressure losses contributed by the fiber bundle in the Affinity Oxygenator were calculated by subtracting the pressure drop across the integrated heat exchanger from the overall pressure drop of the device for each volumetric flow rate.

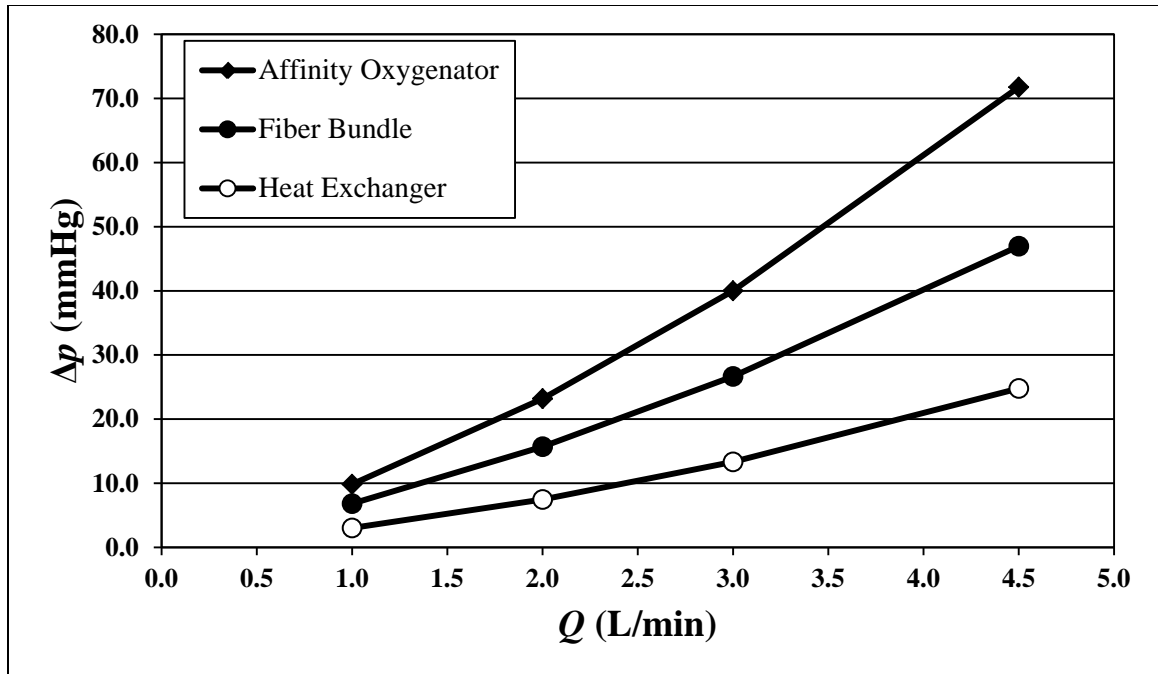


Figure 5.1: Experimental Pressure Drop across Device Components

The pressure drop across the Affinity Oxygenator is a sum of the losses contributed by the fiber bundle and the integrated heat exchanger. Device resistance was measured for flow rates ranging from 1.0 to 4.5 L/min for a Newtonian blood-analogue with a viscosity of 3.3 cP.

5.2. Experimental Perfusion Characteristics

Typically, flow through porous media is precluded from analysis via optical methods due to the opacity of either the solid volume fraction or the fluid media, or both. Without domain-specific experimental data, information regarding internal perfusion characteristics is limited to general assumptions. For example, the superficial velocity is determined analytically based on volumetric flow rate, and therefore assumes a uniform (or homogeneous) flow distribution. However, since the physical velocity is now available from experimental X-ray acquisitions, a more accurate assessment of permeability can be made, accounting for potentially non-uniform perfusion.

5.2.1. Modified Residence Time

Most studies that investigate fluid flow through fibrous media derive permeability equations based on the assumption that all fibers are aligned (oriented either parallel or perpendicular to the flow direction) and experience a uniform flow field.⁶⁰ Following this condition of a uniform, radial flow through the fiber bundle, the average fluid residence time, t_R , is simply

$$t_R = \frac{\Delta x}{\bar{U}_p}, \quad (5.1)$$

where Δx is the fiber bundle thickness; and \bar{U}_p is the average physical velocity. Yet, if the flow is not entirely radial in propagation, \bar{U}_p cannot be assumed to be equal to its analytical value (Eq. 2.27), and therefore t_R remains unknown.

To account for potentially non-uniform perfusion, a modified residence time, t_R' , based on peak-to-peak transit time,¹⁰⁰ is expressed as

$$t_R' = T_N - T_I, \quad (5.2)$$

where T_N is defined as the moment of peak intensity along the outer edge of the fiber bundle; and T_I is the time of peak intensity when contrast first enters the fiber bundle (Figure 5.2).

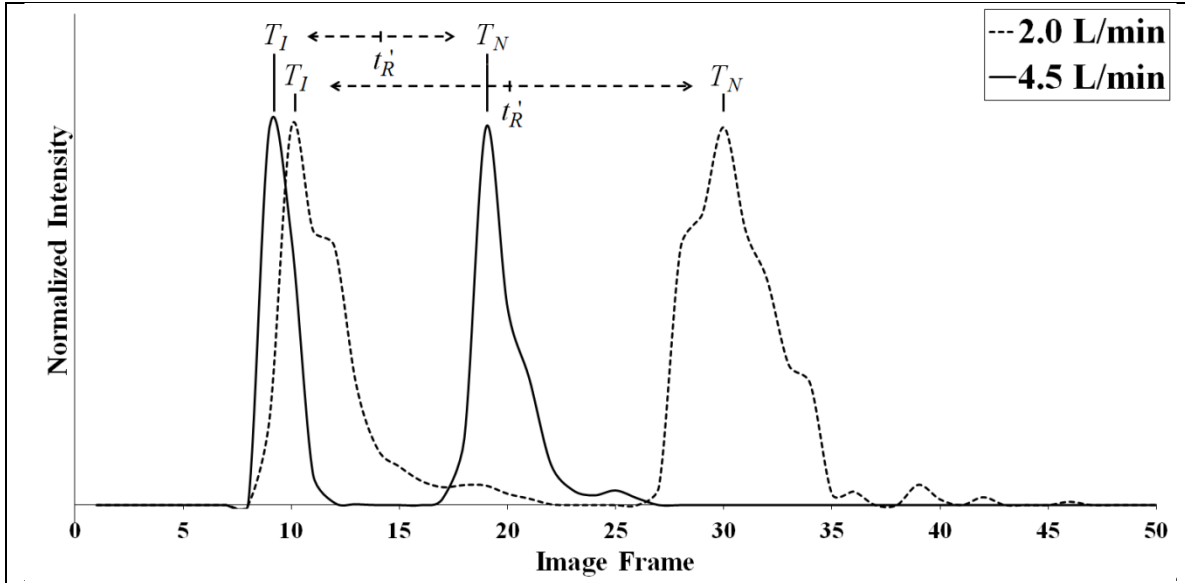


Figure 5.2: Modified Residence Time according to Time-Density Analysis

The normalized contrast intensity curves characterize the time delay between ROIs defined at the inner and outer surfaces of the annular fiber bundle. Curves from the 2 L/min (dashed line) and 4.5 L/min (solid line) flow studies are represented above. Higher volumetric flow rates led to a faster up-stroke, less convective dispersion, and shorter residence time. Image acquisition was 7.5 fps.

The description of t_R' in Eq. (5.2) is effectively a time-density calculation for two ROIs defined at the inner and outer edges of the fiber bundle (cf. §2.6.3.). For the analytical measurements, t_R is significantly longer than the observed experimental data, which necessitated the definition of t_R' in Eq. (5.2). This is exemplified in Table 5.1.

Table 5.1: Calculated Residence Times

Q (L/min)	t_R (s)	t_R' (s)
1.0	8.65	6.53
2.0	4.32	2.73
3.0	2.88	1.87
4.5	1.92	1.27

5.2.2. Experimental and Analytical Fiber Bundle Velocities

Given t_R' , the experimental superficial velocity, U' , was calculated by

$$U' = \varepsilon \bar{U}_p' = \frac{\varepsilon \Delta x}{t_R'} \quad , \quad (5.3)$$

where \bar{U}_p' is the average physical velocity calculated from the time-density analysis. The differences between the analytical (Eq. 2.15) and experimental (Eq. 5.3) evaluations of U are shown in Table 5.2. As would be expected, both U and U' are linearly proportional to the volumetric flow rate.

Table 5.2: Analytical and Experimental Superficial Velocities

Q (L/min)	U (mm/s)	U' (mm/s)
1.0	1.16	1.53
2.0	2.31	3.66
3.0	3.47	5.35
4.5	5.20	7.90

5.2.3. Assessment of Perfusion Direction

The progression of the contrast-enhanced flow through the device at 2 L/min is shown in Figure 5.3. Once the DICOM images had been converted to binary images and spatially smoothed, the cumulative perfusion from previous images was subtracted from each new image, revealing only the net new flow for the given time step (i.e., the current location of the advancing contrast bolus). Flow travels up the inlet tube (Frames 1–3), into the tapering channels (Frame 5), through the fiber bundle (Frames 5–23), into the outer gap (Frame 19), and exits through the outlet port (bottom left of image, Frames 21–23).

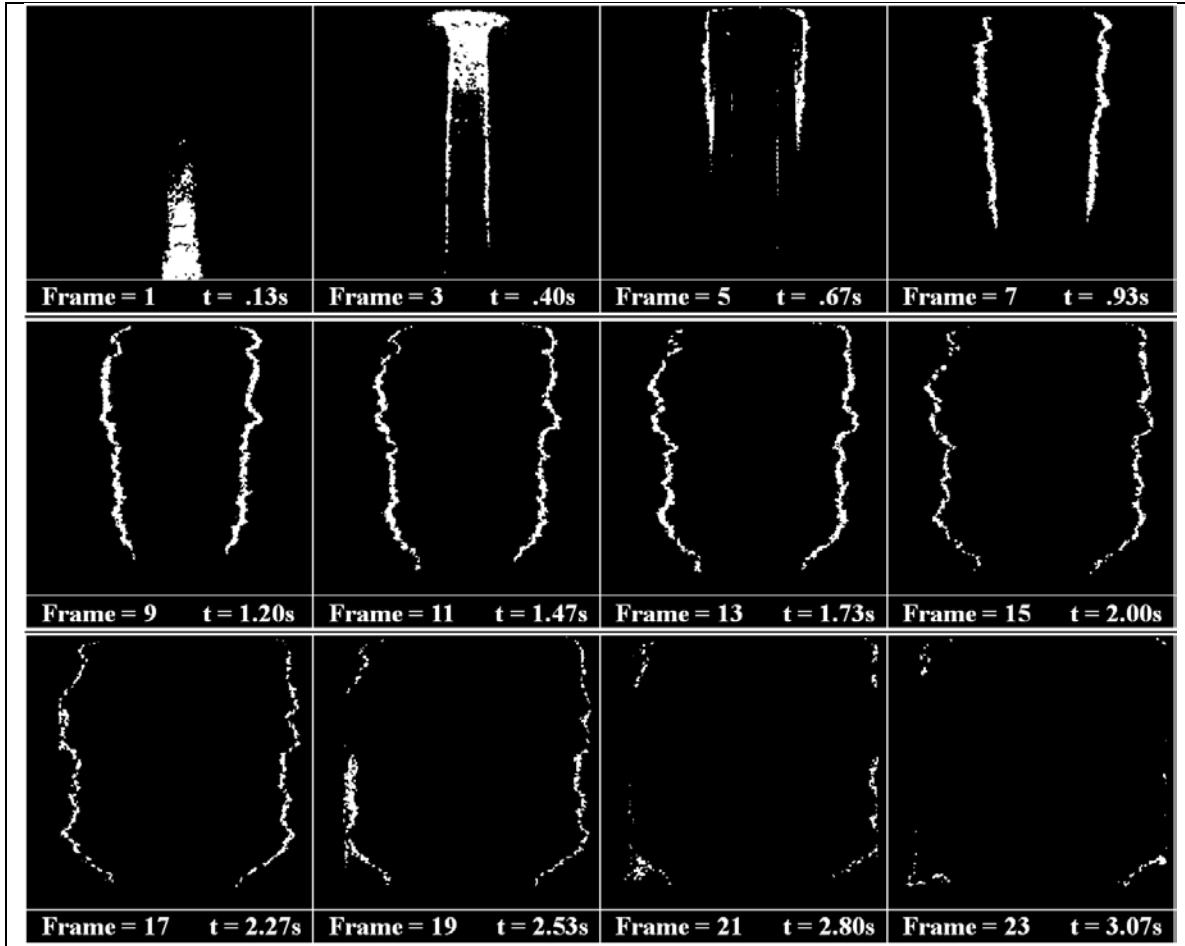


Figure 5.3: Progression of Contrast Perfusion through the Fiber Bundle

The image sequence shows the perfusion of the contrast bolus through the device at 2 L/min flow rate. The device inlet is located at the bottom center of the image frame (see Figure 3.2), and the outlet is positioned at the bottom left. Images were acquired at 7.5 fps. For clarity, every second image in the dataset was omitted in the display above.

Design of the flow channels along with packing density of the fibrous media contributes to the mostly radial perfusion dynamics of the fluid flow. For all of the flow rates tested, no preferential flow was observed due to the position of the outlet port. However, the progression of the contrast wave front is not fully uniform, as is characterized by the uneven distribution of the waveform in the vertical direction. This image series indicates that flow reaches the corners of the device less readily (see specifically Frames 17–23), including the corner closest to the outlet of the device. For the 2 L/min flow rate, most of the flow traveled through the bundle in approximately 2.5 s (Frames 5–23)—visually reflective of t_R' calculated in §5.2.1.

The construction of t_R' is based on a 1-D calculation which has the benefit of allowing the measurement to be partially invariant to non-uniform perfusion contours. In the annular fiber bundle of the Affinity Oxygenator, if flow was indeed single-dimensional (viz., purely radial flow) the average distance traveled by the contrast bolus in each image frame could be used to calculate \bar{U}_p' , as opposed to the time-density techniques employed in Eq. (5.1). Therefore, it might be logical to infer that deviations between the two measurements could yield information regarding the degree of radial perfusion.

Calculations of the average contrast perfusion advancing through the fiber membrane began once all of the dye had cleared the inner flow channels and was entirely contained in the fiber bed. For each orthogonal projection, the left and right-side contrast waveforms were treated as separate measurements. The pixel sum of the contrast in each segment was divided by the vertical height of the fiber bundle to give the average thickness of the waveform if it were compressed to a uniform surface. The physical velocity using the uniform perfusion method (\bar{U}_p'') was averaged for the duration where all of the flow existed in the fiber bundle. Once the perfusion edge reached the outer gap, the measurement was terminated. The averages and standard deviations of the waveforms for each of the flow rates are shown in Table 5.3. In all cases, \bar{U}_p'' was higher than \bar{U}_p' which would suggest a perfusion waveform that is advancing with deflections from purely radial flow. The percent of the flow that moves in a radial fashion is obtained from the ratio between the two measurements.

Table 5.3: Assessment of Radial Perfusion

Q (L/min)	\bar{U}_p' (mm/s)	\bar{U}_p'' (mm/s)	\bar{U}_p' / \bar{U}_p'' (%)
1.0	3.40	4.35 ± 0.00	78.2%
2.0	8.13	9.13 ± 0.11	89.1%
3.0	11.88	12.63 ± 0.03	94.1%
4.5	17.55	19.07 ± 0.12	92.0%

5.2.4. 3-D Perfusion Reconstruction

In addition to providing insight into the 2-D flow through the fiber bundle, the symmetry between each of the outward perfusion waveforms ($n = 4$) in orthogonal planes offers information regarding the 3-D flow field. However, 3-D reconstruction of the biplane dataset was unobtainable using conventional methods such as iterative processing, curve fitting, or stereographic methods due to the dimensions of the fiber bundle and the volume of contrast agent required to maintain a continuous contrast bolus in the presence of fluid dispersion. In the 4 studies, the average maximum depth measurable according to gray-level pixel values (i.e., before binary image conversion) was 22 ± 7.0 mm. This limitation clips the dataset, illustrated in Figure 5.4, thereby yielding no depth information above the maximum gray-level intensity.

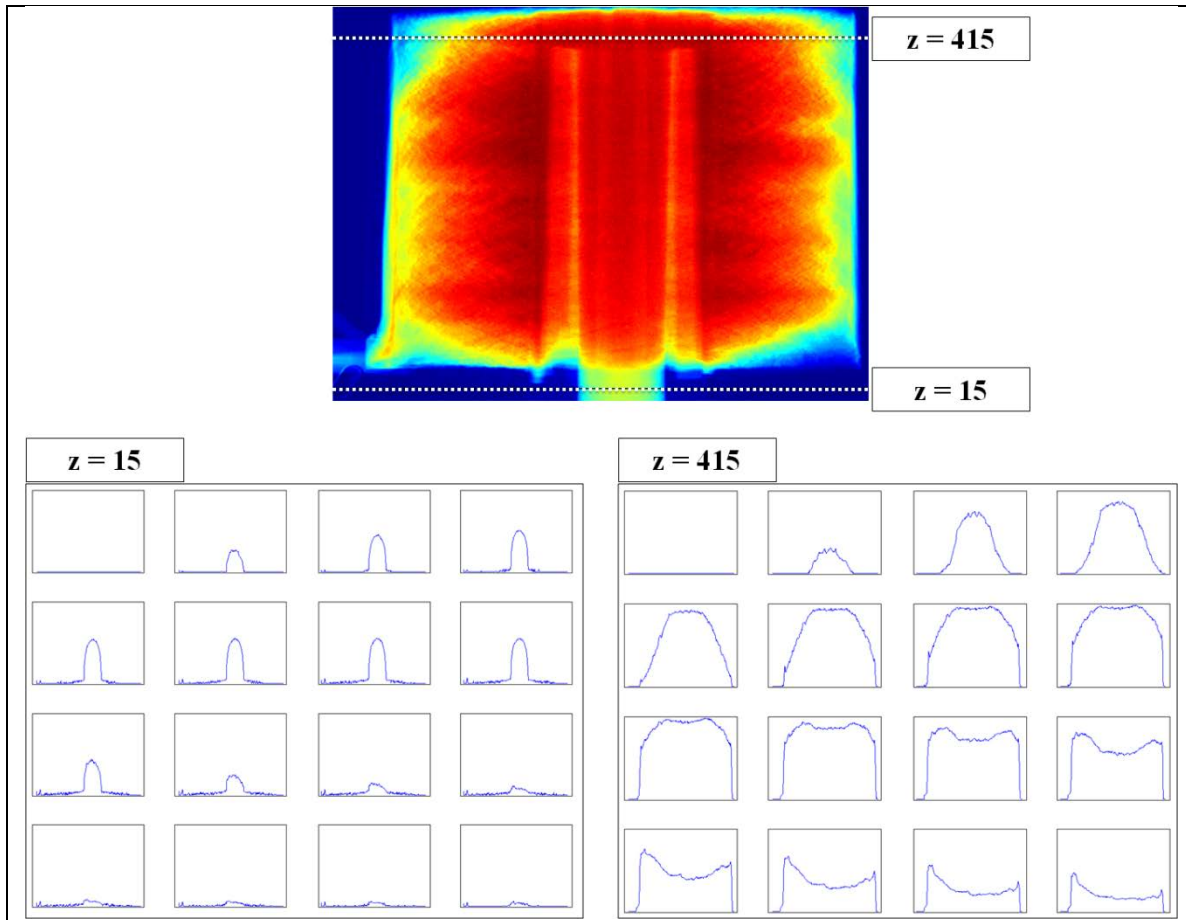


Figure 5.4: Data-Clipping resulting from Maximum Contrast Intensity

Line profiles taken across the inlet tube ($z = 15$) and through the fiber bundle ($z = 415$) illustrate the limitation arising from a maximum contrast intensity. For the z -plane across the inlet tube diameter (15 mm) the cumulative contrast projection is fully characterized by the elliptical intensity curve. Alternatively, the z -plane through the base of the oxygenator housing (diameter is 82 mm) the maximum opacity of the contrast precludes any depth information. In the figure above, the contrast line integrals are displayed at 0.93-s intervals.

The qualitative symmetry observed between the biplane projections justified the approximation of a radial perfusion contour. The outer edge of the contrast bolus in each aligned biplane projection was assumed to be smoothly distributed in the non-orthogonal, z -plane dimension, and the known 3-D position of the device housing was subtracted from the contrast perfusion. Thus, for each row in the aligned projections, the algorithm required exactly two or six reference nodes (see Figure 4.5). Due to the number of assumptions embedded in this formulation, the 3-D volumetric perfusion remains only qualitative. Nevertheless, since the

algorithm interpolates in the non-orthogonal depth of field, the 3-D visualization maintains the same level of quantitative accuracy as the 2-D biplane images, but provides improved spatial clarity to the overall perfusion dynamics (see Figure 5.5).

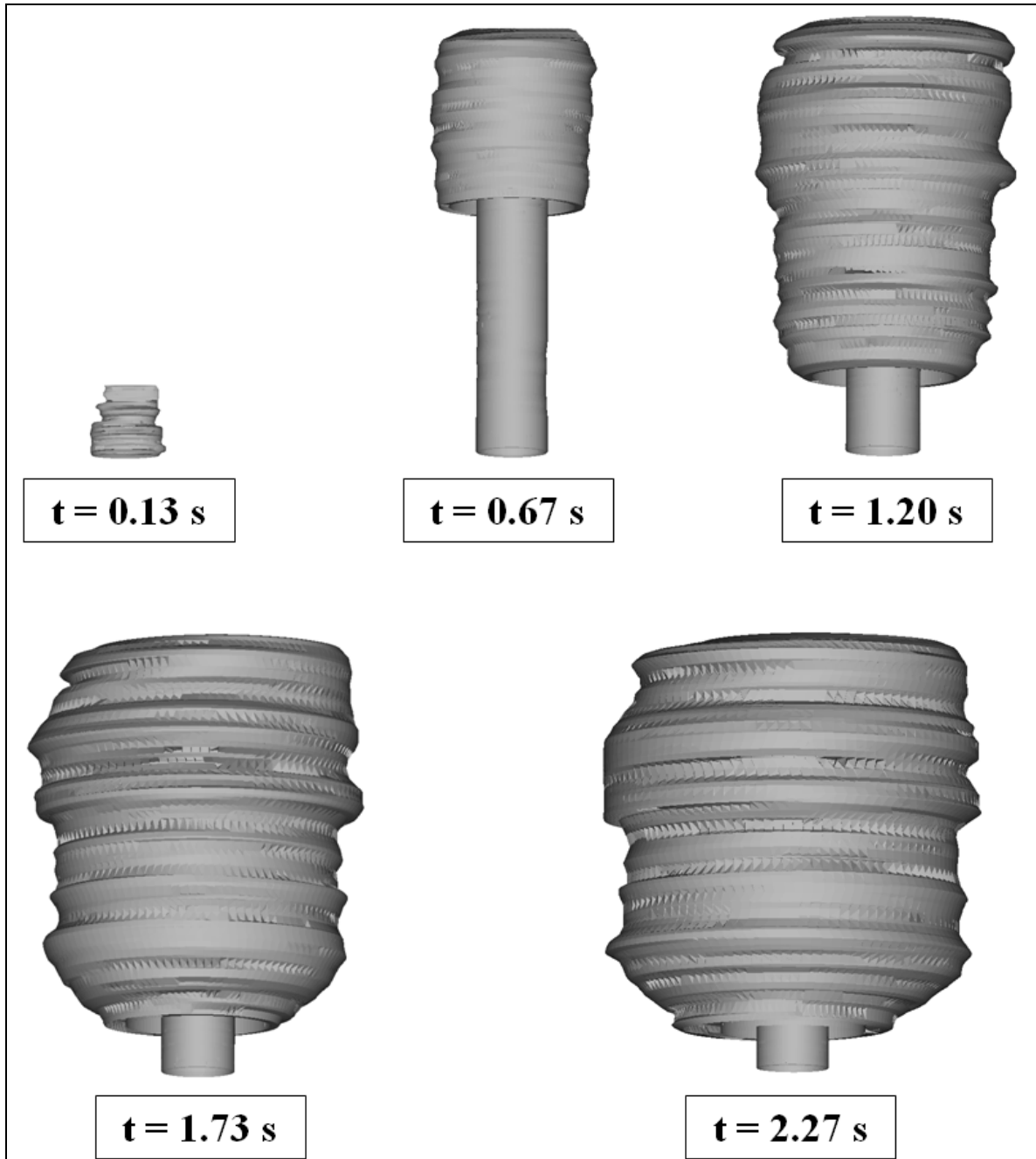


Figure 5.5: Three-Dimensional Volumetric Perfusion at 2 L/min

The outer edge of the biplane projections served as a set of vertices for the generation of a cubic spline in the z-direction. Each spline corresponded to a different slice in the z-plane, whose interpolated values were used to construct a point cloud in 3D. Surfaces were rendered over the point set to give clarity to the volumetric perfusion.

5.3. Permeability Results

In order to compare these results with the empirical results of Ergun⁵⁴, Leva⁵⁵, and others^{71,77}, d_p —whose definition has been somewhat ambiguous (cf. §2.4.6.)—must first be quantified.

5.3.1. Equivalent Spherical Diameter

The coefficients a and b in Eq. (2.20) were obtained from fitting a second-order polynomial to the experimental pressure drop and superficial velocity data. Using the modified Ergun coefficients determined by Macdonald et al.⁷¹ ($\alpha = 180$, $\beta = 1.8\text{--}4.0$), d_p was approximated from Eq. (2.21) as

$$d_p' \cong \frac{29}{25} d_o \approx d_o, \quad (5.4)$$

where d_p' is the experimental spherical diameter; and corresponds to an inertial coefficient $\beta = 2.4$ (from Eq. 2.22). The viscous and inertial coefficients from the referenced empirical data have been shown alongside those found in this study (see Table 5.4).

Table 5.4: Referenced and Experimental Ergun Coefficients

	α	β
Ergun ⁵⁴	150	1.75
Macdonald et al. ⁷¹ (low)	180	1.8
Macdonald et al. ⁷¹ (high)	180	4.0
Leva ⁵⁵	200	1.75
Experimental Data	180	2.4

5.3.2. Analysis of Empirical Coefficients

The permeability of the fiber bundle according to the pressure losses measured across the device and the superficial velocities derived from \bar{U}_p'' are shown in Figure 5.6. For reference, the permeability calculated using the empirical coefficients of Ergun⁵⁴, Leva⁵⁵, and the range by

Macdonald et al.⁷¹ have also been graphed. The measured permeability of the device corresponds well with the data of Leva and the lower permeability range determined by Macdonald et al., though the experimental data showed lower permeability and greater inertial resistance than those of the conventional Ergun coefficients. Nevertheless, experimental permeability measurements were relatively consistent with the empirical data, which would be expected based on the agreements typically seen in CFD simulations.^{29,30,37-42}

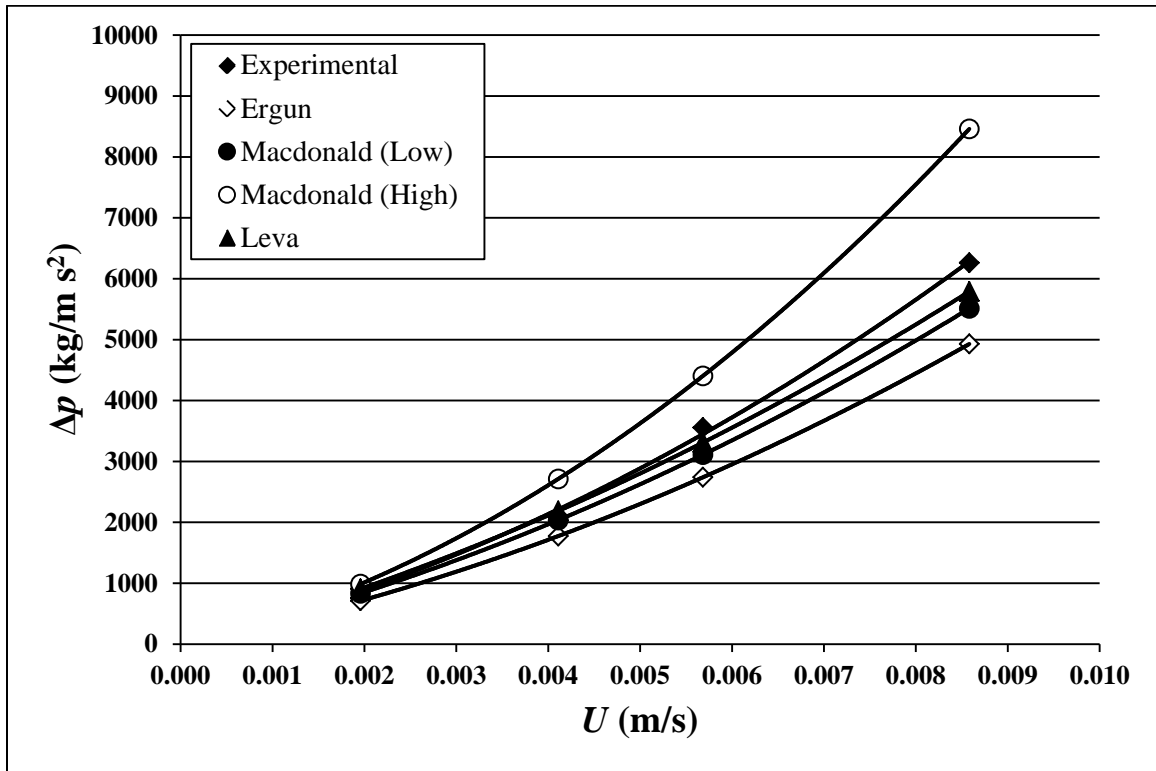


Figure 5.6: Fiber Bundle Permeability

Pressure losses calculated using the empirical Ergun coefficients of Macdonald et al.⁷¹, Leva⁵⁵, and Ergun⁵⁴ have been plotted alongside the experimental pressure measurements (dashed line) according to the experimental superficial velocity.

5.4. Discussion

Validation of the numerical flow simulations require assessments of the fiber bundle permeability to be accurately characterized. Early empirical studies laid the foundation for today's understanding of the viscous and inertial forces influencing flow through porous media.

For slow flows, the Darcy permeability describes resistance (i.e., pressure losses) to flow in a linear relationship to the superficial velocity. As the velocity increases, momentum carries fluid particles away from their streamlines and the inertial effects result in an increased flow resistance.

Conventionally, permeability is characterized by the pressure losses corresponding to the superficial velocity. Yet, the superficial velocity does not explicitly describe the features of the porous medium, including the porosity and packing arrangement. In fact, U is prescribed with $\varepsilon \equiv 0$, therefore U portrays the volumetric flow rate as a uniform velocity field through the device. When the porosity of the fluid zone is either randomly or isotropically distributed, and if the container is large enough such that wall effects are negligible, a uniform superficial velocity is generally an acceptable assumption. Yet, when perfusion is non-uniform or $\varepsilon_{eff} \neq \varepsilon$, other measures are required.

For the investigation above, the motivation for defining U based on t_R' was due to the observed non-uniformity of the velocity wavefront in the imaging acquisitions. Since the angiographic images had been reduced to a binary flow field, the single-dimensional approximation of the time-density method was valid for ROIs defined at the entry and exit regions of the fiber bundle (cf. §2.6.3.). The only requirement of t_R' is that the fluid perfusion is rotationally symmetric; this was verified according to the similarities of the fluid propagation in the orthogonal projections (Table 5.3).

In order to determine the degree of radial perfusion throughout the fiber bundle, a ratio was established between the experimentally-derived physical fluid velocity and the average thickness of the contrast bolus per time step. The method showed that as the flow rate increased, the experimental velocity approached the velocity of the net new contrast in each image frame. This may suggest that at higher flow rates (e.g., $Q \geq 2.0$ L/min), the velocity direction through the fiber bundle in the Affinity Oxygenator travels predominantly in a radial fashion.

6. EVALUATION OF NUMERICAL DATA

6.1. Flow Characteristics

The results from the CFD simulations correlated very well with the numerical data obtained by Zhang et al. for a 5 L/min inlet flow rate through the Affinity Oxygenator.⁴⁰ The pressure distribution and flow characteristics from the numerical predictions were proportionally consistent for all inlet velocities tested. In the following sections, all visualizations will be represented by a clinically relevant 2 L/min flow rate. Perfusion through the device is depicted by the pathlines illustrated in Figure 6.1. Blood flows up the inlet tube and reverses direction in the inner flow channels. The tapered channels provide no resistance to the flow, which allows the blood flow to fill the entire inner radius of the fiber bundle. The pathlines generated in the CFD simulations show that the flow through the fiber bundle is mostly radial, and the position of the outlet port does not appear to influence the perfusion of the flow more favorably towards the device outlet. This was consistent with the DSA perfusion profiles, whereby the orientation of the outlet port did not affect the uniformity of the radial distribution for any of the flow rates ranging 1–4.5 L/min.

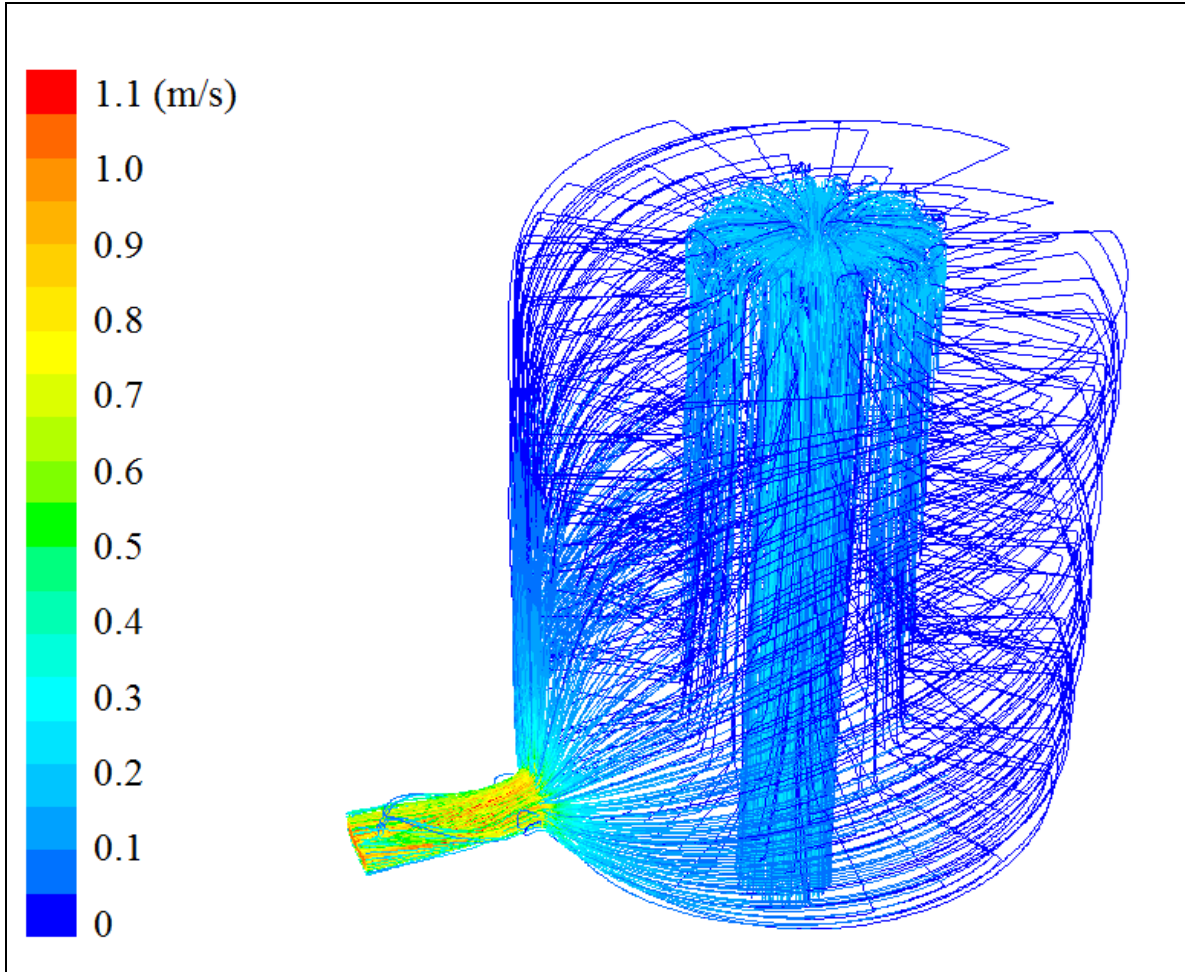


Figure 6.1: CFD Pathlines of Fluid Flow

Three-dimensional pathlines through the fiber bundle are shown according to velocity magnitude (at 2 L/min). The pathlines illustrate flow through the inlet tube (bottom-middle of figure), reversal of direction in the inner channels, and radial perfusion through the fiber bundle. Once flow reaches the outer gap it circulates towards the device outlet (bottom-left).

Once the blood flow has permeated through the fiber bundle, it is collected in an outer gap (3 mm thickness) and circulates to the outlet port. Upon exiting the device, the flow speed is accelerated, as depicted by the reference colorbar.

6.2. Pressure Distribution

The CFD pressure distribution across the device characterizes a concentrically uniform pressure gradient throughout the fiber bundle (Figure 6.2). Most of the pressure losses across the

device are due to viscous and kinetic forces in the fiber bundle, as the pressure in the inlet tube and outer gap are relatively uniform. The outer gap facilitates the circumferentially uniform flow pattern since the pressure is approximately equal in the spacing between the fiber bundle and device housing. The concentric pressure gradients account for the even radial perfusion seen in the bench studies (cf. §5.2.3.).

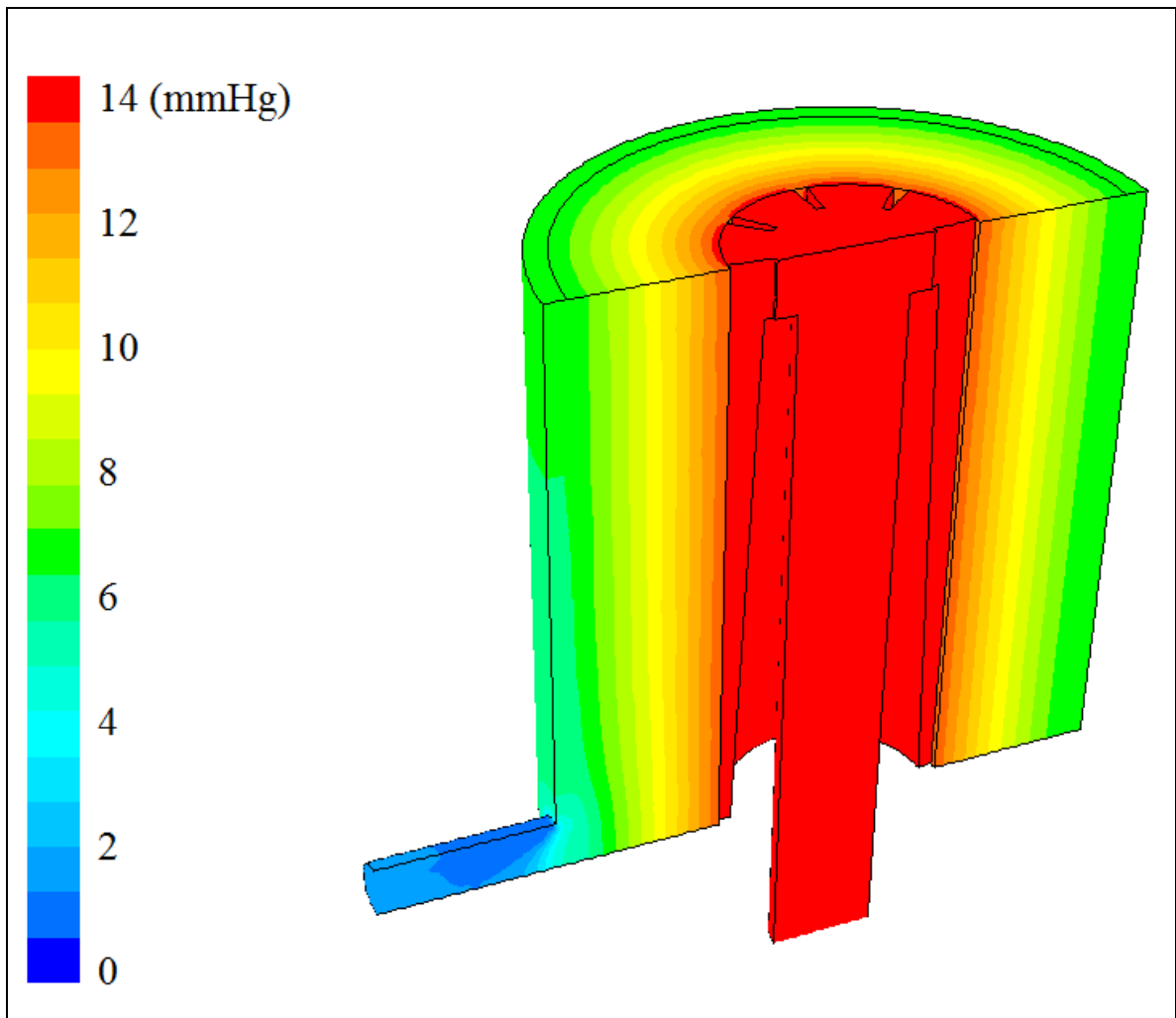


Figure 6.2: CFD Device Pressure Drop

The bulk of the resistance downstream of the integrated heater occurs across the fiber bundle and is characterized by a concentric pressure gradient through the annular fiber bundle. The figure represents the pressure drop for a 2 L/min volumetric flow rate.

6.3. Velocity Profile

The magnitude of the velocity vectors through the fiber bundle cross-section and outer gap is shown in Figure 6.3. The image cross-section was obtained from the middle of the fiber bundle height. The vectors indicate a highly radial flow direction, correlating well with the concentric pressure contours from Figure 6.2. As flow enters the fiber bundle (radial-outward), small eddies are seen behind the support columns of the inlet tube, but recover within the first few millimeters of the fiber bundle. Once through the fiber bed, flow is collected in the outer gap and accelerates towards the outlet port (orientated at the top of the image).

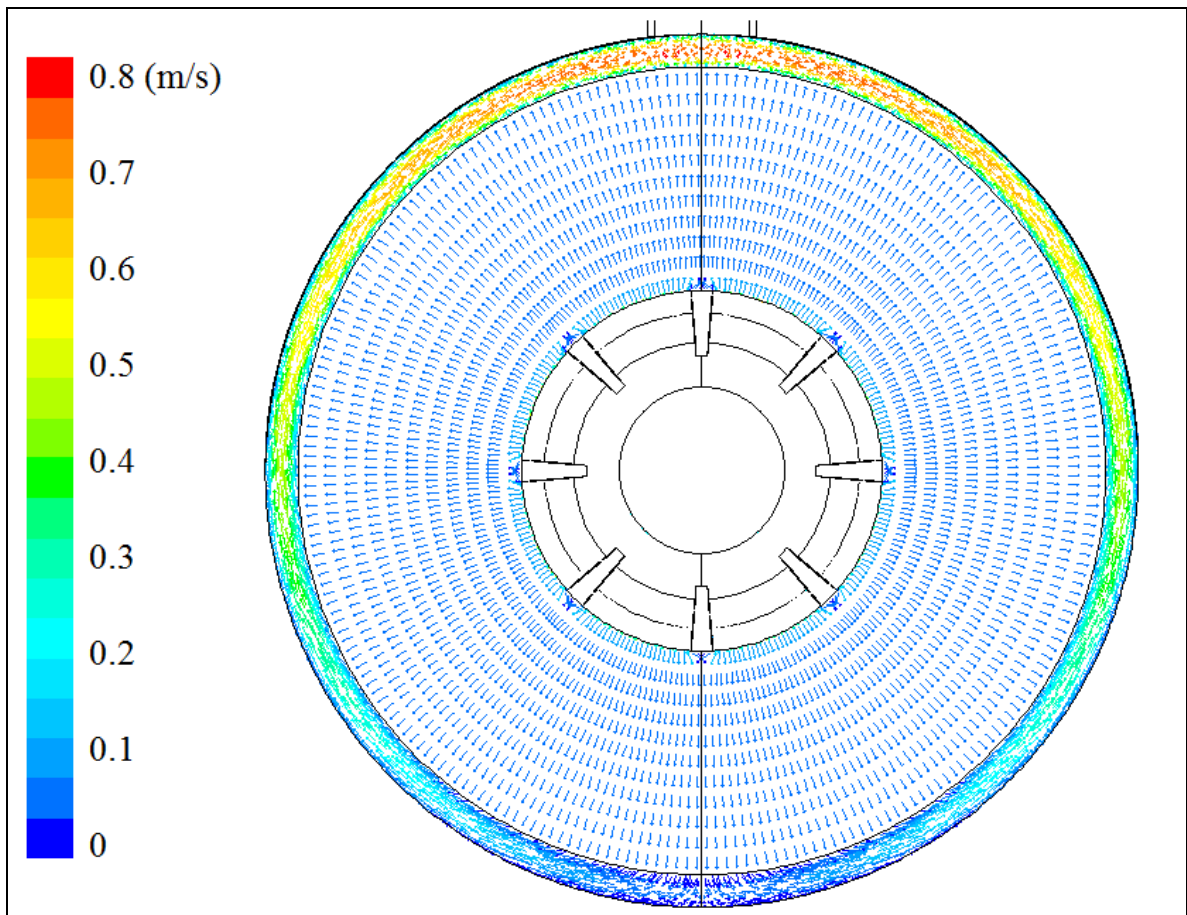


Figure 6.3: CFD Velocity Vectors on Fiber Bundle Cross-Section

Velocity vectors in the fiber bundle (physical velocity) and outer gap at 2 L/min. Vectors have been prescribed with uniform length, where the magnitude is indicated by the reference colorbar. Cross-section was at 50% of the fiber bundle height.

The computational and experimental (MCC) velocity vectors for corresponding planes are shown in Figure 6.4. Both images display *physical* velocities; where the magnitude in the CFD image is characterized by a color scale, and the MCC vector magnitudes according to its vector length. Five representative points have been taken to compare the two measurements: two sampled in the open channels of the device, and three points from various sites within the fiber bundle. The vector magnitudes in the MCC figure are the averages of nine vector magnitudes (center plus eight-nearest grid points) surrounding the sample location in the corresponding CFD image.

In portions of the device preceding the fiber bundle (inlet tube and inner flow channels), the velocity values calculated for the 2 L/min flow by the CFD and MCC methods are comparable. However, in the fiber bundle, where the Ergun approximation was utilized to characterize the fiber membrane, the CFD velocity measurements severely underestimated the experimental velocity, and approximated values consistent with the analytical values of U .

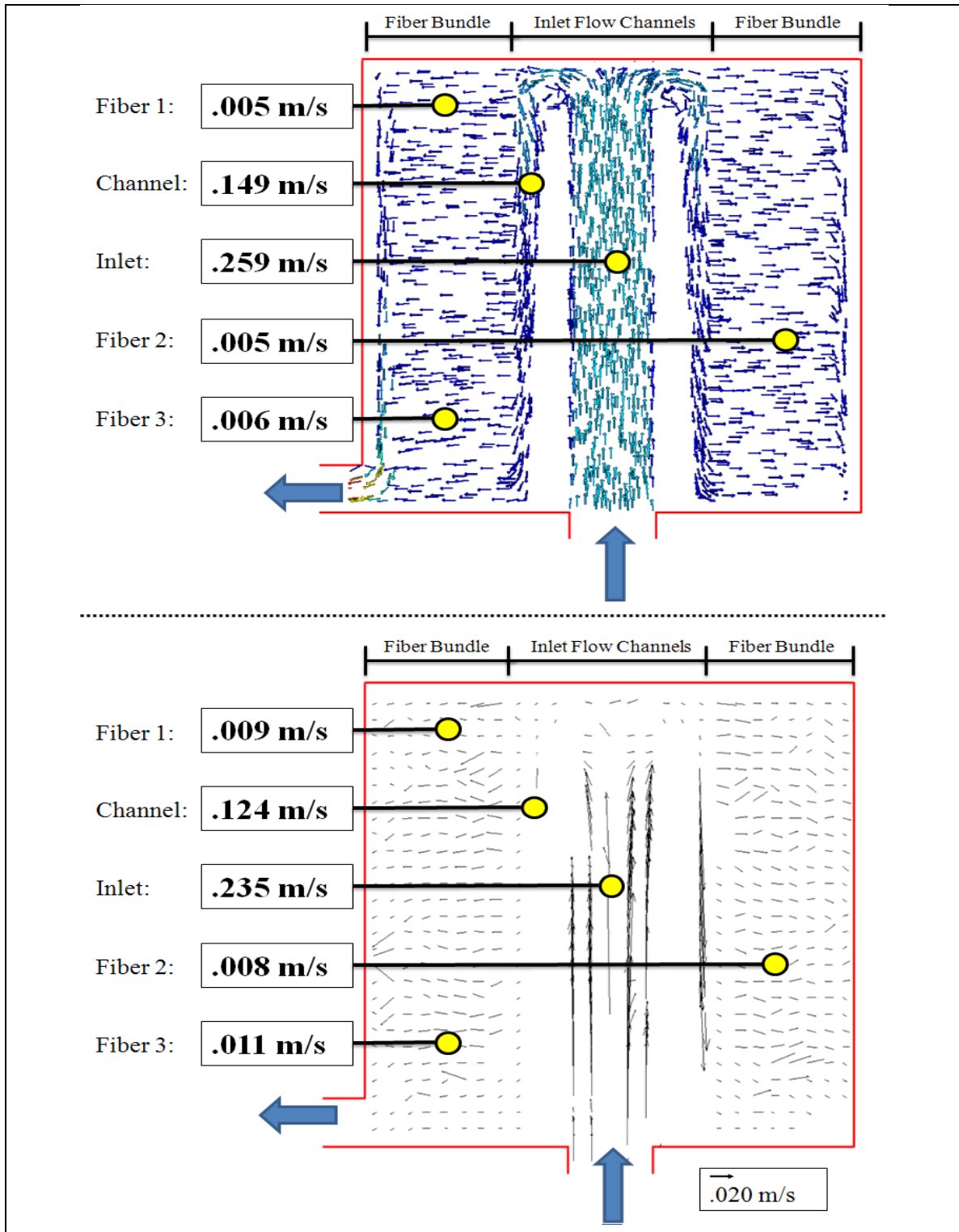


Figure 6.4: Numerical and Experimental Velocity Vectors at 2 L/min

The velocity vectors calculated by the CFD study upstream of the fiber bundle are consistent with those measured with the MCC technique. Yet, according to the porous media assumption, the numerical analytic studies significantly under-predicted the velocity within the fiber bundle.

6.4. Improvements to the Numerical Models

The numerical studies correlated well with the experimental data in the portions of the device preceding the fiber bundle. However, the porous media approximation significantly under-predicted the velocities in the fiber bundle. Observing that the discrepancy between the numerical and experimental data was confined to the fiber bundle, specifically $\bar{U}_p < \bar{U}_p''$, a scaling factor was added to modify the fiber bundle characteristics. Therefore, in order to fit the numerical solutions to the experimental data and account for the non-uniform flow behavior, a scaling factor, ω , defined as the ratio between the experimental uniform *physical* velocity and the analytical *physical* velocity, was defined as

$$\omega = \frac{\bar{U}_p''}{\bar{U}_p}, \quad (6.1)$$

and

$$U = \frac{\varepsilon}{\omega} \bar{U}_p'' . \quad (6.2)$$

For the 1–4.5 L/min studies, ω was calculated to be 1.69 ± 0.06 . CFD simulations were re-run with a prescribed porosity of 0.27 (ε/ω), while using the same permeability and inertial coefficients previously determined. The magnitudes of experimental MCC velocity vectors, along with the numerical results calculated using both the analytical porosity and the modified porosity are listed in Table 6.1.

Table 6.1: Referenced Physical Velocity Values from Experimental and Numerical Studies

Q (L/min)	\bar{U}_p (mm/s) at References Sites in Figure 6.4					
	Fiber 1	Fiber 2	Fiber 3	Inlet	Channel	
1.0	4.4	5.2	5.4	127.0	72.8	Experimental
	2.7	2.5	2.9	141.0	75.0	CFD, $\varepsilon = 0.45$
	4.8	4.2	4.9	140.0	76.0	CFD, $\varepsilon/\omega = 0.27$
2.0	9.2	8.3	10.9	235.4	124.3	...
	5.4	5.0	5.8	258.5	148.5	
	9.5	8.6	9.9	258.5	148.5	
3.0	14.2	13.6	14.8	281.5	241.6	
	8.2	12.4	9.0	374.4	241.8	
	14.0	12.6	15.5	374.4	241.8	
4.5	21.6	19.6	24.7	523.9	365.5	
	12.3	11.1	13.8	552.0	336.0	
	21.0	19.0	24.0	539.0	341.0	

The average absolute difference in the fiber region between the existing and modified numerical techniques improved from $44 \pm 4\%$ to $6 \pm 5\%$. At the same time, the modification did not affect the absolute difference in the non-porous zones; both simulations resulted in $11 \pm 11\%$ absolute differences in experimental and numerical calculations. This is further illustrated in Figure 6.5.

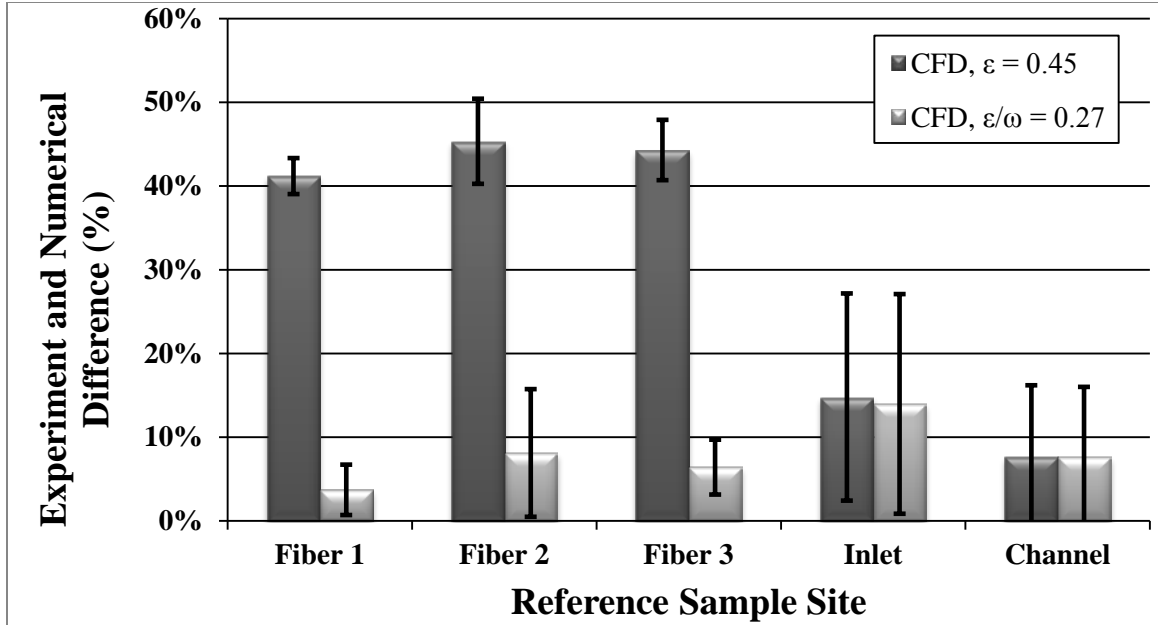


Figure 6.5: Absolute Difference between Experiment and Numerical Velocity Values

The modified porosity factor decreased the average difference between CFD and experimental (MCC) velocity values in the fiber bundle region of the device from $44 \pm 4\%$ to $6 \pm 5\%$. No change was observed in the regions preceding the fiber bundle (i.e., non-porous media zone).

The modified porosity does not significantly affect the computed pressure losses as shown in Figure 6.6. At lower flow rates (1–3 L/min), the numerical data under-predicted the fiber bundle resistance, while simulations employing d_p' obtained the closest correlation to experimental pressure drop. Use of the Ergun terms with $d_p = d_o$ were similar to the results from d_p' , as expected. The figure also indicates, for global pressure losses, the model using Ergun terms according to the fiber hydraulic radius (Eq. 2.25) calculated a much lower fiber bundle resistance than shown experimentally.

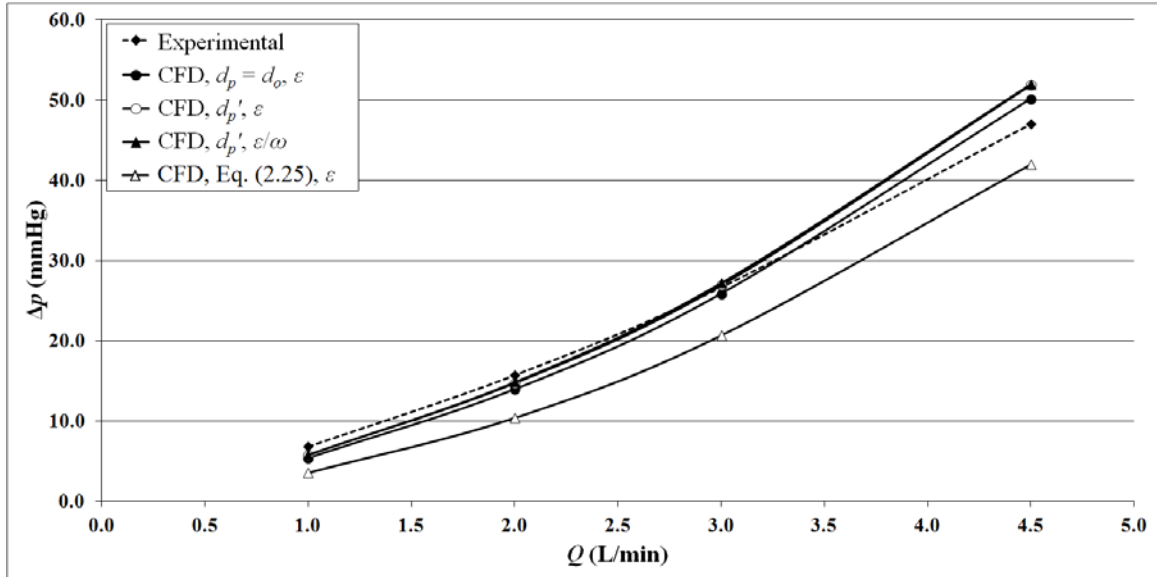


Figure 6.6: Evaluation of Numerical Parameters

Comparison of the pressure losses calculated using the empirical Ergun coefficients, varying values for d_p and ε .

6.5. Discussion

The porous media model functions as an added momentum sink in the governing momentum equations according to an empirically determined flow resistance. Since the solid fraction of the porous medium (viz., the fibers themselves) is not physically represented in the CFD model, steady-state simulations of single-phase flow through isotropic porous media using numerical solvers such as ANSYS Fluent perform calculations according to the analytical superficial velocity, and effectively, incorporate the porosity during post-processing.⁷⁰

Overall, the model captured the perfusion dynamics reasonably well when compared with the experimental X-ray image sequence. The radial flow behavior was reproduced numerically and the computed global pressure losses agreed with measured values. However, the magnitude of the velocity vectors was significantly under-predicted in the computational studies. Although this condition is usually present at higher Re , the various assumptions traditionally employed in

calculations of blood flow through membrane devices may result in similar errors. In fact, it can be shown that pressure fields computed in this way are least accurate at low Re .⁴⁶

Since the construction of ε/ω improved correlation between experimental and CFD studies, the lower values originally obtained from the numerical studies could be explained by factors that affect the porosity and/or the physical velocity. A lower porosity could be reflective of flow channeling (shunt) in the fiber bundle or the condition of $\varepsilon_{eff} \ll \varepsilon$; though the latter is less likely at higher flow rates. Woven fiber bundles and greater membrane rigidity result in more homogeneous structures, but high blood-side pressures could lead to compressibility of the porous media and deformation of the fiber bundle.¹²⁷ Thus, when permeability is defined according to porosity and other analytical parameters, it is unaffected by the level of fiber bundle compressibility or fiber orientation.^{60,61,68,128,129}

By introducing a scaling factor into the porosity value, the average absolute difference between the experimental and numerical velocities was reduced from 44% to 6%, without affecting the pressure calculations or radial flow behavior. Due to the formulation of the porous media model, maintaining the empirical permeability and inertial coefficients, while prescribing a reduction in void space, equates to a higher velocity magnitude, yet unchanged pressure distributions and perfusion characteristics. The correction therefore does not influence computation of the overall conservation equations, but acts as a further post-processing step. However, this modification has not been tested in simulations that incorporate porosity into the conservation equations, such as multiphase and reaction models.

Although directly modeling individual fibers in a large-scale 3-D model is impractical with present-day computing, little work has been done to evaluate the validity of the porous media approximation outside of overall pressure contours. Recently, Mazaheri and Ahmadi simulated 2-D fluid flow through a membrane oxygenator with 60% porosity at 4 L/min and compared the porous media assumption with a model of individual fibers. This study showed that the porous media model did not accurately characterize the non-uniform flow distribution.³⁹

Interestingly, their results indicated nearly a six-fold increase in velocity magnitude when *using* the porous media approximation when compared to the direct fiber modeling;³⁹ this is opposite what was shown here, experimentally, possibly due to restrictions in 2-D modeling.

When comparing blood flow velocities between the numerical approximations and experimental data, it is important to recognize that the velocities observed experimentally are the *physical* fluid velocities. Yet, most studies that characterize CFD flow properties in the fiber bundle report the *superficial* velocity (which does not account for fiber bundle porosity). For example, Funakubo et al. reported velocity values approximately 2–6 mm/s for a 3 L/min flow rate through a fiber bundle with 40% porosity.³⁰ Likewise, at 5 L/min volumetric flow through a 25% porous fiber bundle, Sato et al. determined fiber membrane velocities ranging 7–8 mm/s.⁴¹ Similarly, Zhang et al. calculated velocities of 5–6 mm/s for a 5 L/min flow through a fiber bundle with 45% porosity.⁴⁰ However, without distinguishing between superficial and physical velocities, there exists the potential for misinterpretation of the data. For instance, according to the work by Gartner et al., which correlated thrombus deposition in a fiber membrane with low CFD velocity values²⁹, assertions regarding a minimum flow speed “threshold” are significantly different if determined based on the superficial, versus the physical, velocity.

The experimental velocities were measured using a cross-correlation method based on binary patterns between sequential image frames. Unfortunately, the MCC technique in its simplest construction does possess a degree of subjectivity in regards to definitions of the template and search window sizes, and is therefore commonly validated with other forms of measurement—such as thermal imaging. However, in these previous studies, supplemental experimental data were required because efforts to identify a non-subjective means for establishing correlation cut-off values were not identified.¹²⁴ Alternatively, the convergence tests and approach performed in §4.2.3. have likely reduced user biases.

7. APPLICATIONS AND LIMITATIONS

Results from the biplane DSA study revealed a non-uniform perfusion profile and significantly higher fluid velocities than previously shown with traditional numerical models. These findings and the techniques developed in this body of work extend to numerous applications and study principles. Some of these are addressed below, followed by a brief discussion on the limitations of the study protocol.

7.1. Clinical Impact

Blood permeating through the fiber bundle experiences varying levels of shear stress due to the complex flow paths around the outer surfaces of the fibers. For incompressible flow, Cook et al.¹³⁰ presented a root-mean-square average shear stress, $\bar{\tau}$, for flow through any porous medium as

$$\bar{\tau} = \sqrt{\frac{\mu \Delta p Q}{V_v}}. \quad (7.1)$$

For Newtonian approximations of blood, the shear rate, $\dot{\gamma}$, is directly proportional to shear stress by the fluid viscosity:

$$\dot{\gamma} = \frac{\bar{\tau}}{\mu}. \quad (7.2)$$

Both $\bar{\tau}$ and $\dot{\gamma}$ are important parameters in evaluating device-imparted blood trauma such as platelet and leukocyte activation^{130–132}, thrombosis^{132–134}, hemolysis¹³⁵, and interactions between cellular constituents^{136,137}. These values, along with Re_i (according to \bar{U}_p' and d_p') have been reported in Table 7.1 for all flow rates tested.

Table 7.1: Reynolds Number, Shear Stress and Shear Rate

Q (L/min)	Re_i	$\bar{\tau}$ (dyne/cm ²)	$\dot{\gamma}$ (s ⁻¹)
1.0	0.39	5.89	178.5
2.0	0.83	12.64	383.1
3.0	1.15	20.17	611.2
4.5	1.73	32.79	993.7

The development of thrombosis in membrane oxygenators is a serious complication which results in decreased gas exchange efficiency and increased device resistance. Both intrinsic design characteristics and extrinsic (or systemic) factors contribute to the formation of thrombotic deposition; however, the exact mechanisms remain unclear. Studies by Goodman et al. suggest that a combination of surface and flow properties contribute to the *initiation* of thrombus formation, whereas thrombus *growth* is primarily a function of the flow dynamics.¹³⁴ Runyon et al. showed that surface-bound inhibitors play a significant role in regulating clotting propagation in structures with small volume-to-surface ratios, but were less effective in large volume-to-surface channels where shear forces became the dominant factor; though these results may be system-dependent.¹³³ Alternatively, thrombosis may be initiated by circulating emboli (e.g., atherosclerotic debris, calcium, air, fat, platelet thrombi, etc.) that become trapped in the fiber bundle and physically alter the normal fluid path creating pockets or areas of stagnancy.¹³⁸ Thus, the fluid dynamics of the system play a critical role in thrombus development and growth.

Some of the most frequently correlated properties of fluid flow with thrombotic sites are recirculation, low blood flow/mass flux, and low shear rates.^{29,133,134,139} Gartner et al. created frequency maps of clinical thrombotic deposition in a fiber bundle cross-section and observed an association between sites of thrombosis and low blood velocities predicted by numerical studies. Without making specific claims, they postulated that thrombosis deposition maps, correlated with CFD simulations of blood flow, could be used to prescribe a critical thrombosis threshold, say 4 cm/s.²⁹

Similarly, Rayz et al. noticed a strong similarity between slow-flow regions predicted by CFD simulations and intracranial aneurysm thrombosis in vivo, and concluded that areas of low velocities and low wall shear stress are particularly prone to thrombus formation. Though careful not to assert that flow stagnation is the primary cause of thrombus formation (otherwise clotting would immediately form in the smaller arterioles, capillaries, and veins), the numerical simulations employed a minimum velocity threshold of 10% of the normal velocities observed in the proximal and distal vessels (approximately 1.5–2.5 cm/s) as a metric for thrombus development.¹³⁹

Since a clinically relevant threshold value has not yet been described, numerical simulations could be useful in defining (or at least, qualifying) such a value. However, it has now been shown that the scale of these numerical approximations in fiber bundles is inaccurate. The techniques developed through a simple X-ray imaging system provide a more realistic measure of the physical velocity through membranous devices and can be used to improve numerical results.

Compatibility of membrane oxygenators in the extracorporeal circuit have been shown to be more favorable with modified fiber surfaces (e.g., heparin-coated).³² Yet, studies by Lehle et al. reveal that despite heparin-coatings or continuous low-dose anticoagulant infusions, about 10% of membrane oxygenator failures are due to thrombotic deposits in the blood path.²⁴ Once thrombosis has initiated, the clot itself becomes the greatest initiator for further thrombus development. In addition to posing significant risk to the patient from potential thromboemboli, thrombus growth alters the *natural* fluid dynamics of the device, further augmenting thrombotic deposition. The experimental techniques developed in this research effort, while useful for evaluating characteristics such as low velocity, recirculation sites (eddies), local mass flux, and the effects of gravitational orientation, would also allow investigation into the fluid dynamics *post* thrombus development. Since X-ray DSA angiography requires very low-dose X-ray exposure, the extent and dynamics between blood flow and thrombotic deposition can be assessed in situ.

The series of angiographic image projections shows the central region of the fiber bundle experiencing higher flow velocities, with the corners of the device being perfused less readily—sites that may be prone to thrombus formation following extended use. If thrombosis develops, the physical characteristics of the altered flow field could easily be obtained with X-ray DSA methods. Assessing patterns of thrombus development would allow a direct correlation between convective flux and fluid velocities observed experimentally. Further investigation is required to compare oxygenator flow characteristics with clinical endpoints such as thrombus deposition.

7.2. Application for Improving Device Performance

One of the most important design features of an artificial lung is uniform flow perfusion. Flow path inhomogeneities can lead to stagnant blood flow and thrombosis, and flow shunts reduce efficiency of the device. Although studies that analyze flow around individual fibers are useful for assessing optimal packing arrangements based on local mass transfer, fiber resistance, and shear stresses^{73,140,141}, these results do not translate into global perfusion dynamics such as geometrically-induced eddies or regional mass flux. Furthermore, most CFD applications in membrane devices investigate steady-state conditions such as pressure distribution and velocity fields, which are useful for predicting overall device-imparted blood trauma and relative perfusion behavior; though, much information could be gained from an investigation into the transient nature of the blood as it enters and leaves the device.

Studies like those performed by Funakubo et al. and Asakawa et al., in which a series of design constraints are tested, could benefit greatly from a transient visualization of deoxygenated blood as it enters the device rather than inferring a uniform distribution from pressure contours.^{30,38} Similarly, Graefe et al. attempted to optimize flow uniformity by testing various inlet and outlet port designs according to particle residence time, but this approach yields no information regarding the distribution of flow in the fiber bed.³⁶ Others, like Hirano et al., have experimentally investigated the time-dependent blood perfusion in a fiber membrane in attempt to

improve flow uniformity through the measurement of changes in electrical potential across electrodes arranged in the fiber bundle, but the spatial resolution was limited and the technique required physical (destructive) alterations to the oxygenator for analysis.⁵²

The insight gained from of a transient investigation of fluid flow extends to other membranous devices as well. For example, hollow-fiber hemodialyzers are designed to achieve uniform flow distribution in order to eliminate significant blood-to-dialysate flow mismatches, and optimize diffusion efficiency.^{43,44} With the experimental protocols established using biplane X-ray DSA, researchers can now observe filling effects in the presence of geometrical configurations or environmental influences real-time and without requiring destructive adjustments to the device.

7.3. Limitations of the Study Protocol

The biplane DSA image sequence provided valuable insight into the fluid dynamics of the artificial lung device and easily captured the fluid perfusion at volumetric flow rates up to 4.5 L/min. Although propagation through the fiber bundle at the highest flow rate lasted for approximately 10 frames, higher temporal response would improve flow field resolution allowing more accurate characterization of the perfusion behavior and minimize errors in tracking algorithms.

Image distortions arising from a point-source X-ray beam may necessitate preprocessing modifications to the projection data set, although new flat-panel detectors and internal calibrations can help minimize such spatial errors. Further, the oxygenator was placed in the center of the X-ray image field to reduce parallax between the biplane projections during the present studies. Occasionally, alignment of biplane projections did require rotation of the image $\pm 1\text{--}2^\circ$ about the z-axis. Based on known dimensions of the oxygenator, distortions were measured to be less than 8%, and therefore no adjustments were made to reshape the digital X-ray images.

Most calculations did employ a priori information of the flow field or make assumptions based on the annular shape of the oxygenator. These were justified by noting a high level of symmetry between the biplane projections and assuming homogeneous contrast diffusion in the bulk fluid. Although these assumptions were believed to be reasonable, parameters of the flow field could be better approximated using accompanying projections (e.g., additional rotational or through-plane projections). Reducing the domain to a binary flow field minimized the complexity of the tracking algorithms, but may have contributed to greater error in the calculations.

The MCC method has been shown to be a reliable tool for measuring fluid flow, but does possess a degree of subjectivity in regards to definitions of the template and search window sizes, and therefore requires complementary methods for validation. Therefore, convergence tests were conducted to limit user biases. Computationally, the cross-correlation technique utilizes a brute force method, so it is not largely efficient, and further, the MCC method is not invariant to image scaling or rotation, which may limit its functionality in other applications.

Fiber bundle perfusion is typically perceived as laminar in nature^{29,30,37-42} due to previous estimates of low Reynolds numbers ($Re < 10$) through the fiber bundle at flow rates as high as 5 L/min.^{30,73,140,141} Interestingly, $Re_i < 1.75$ was calculated despite the observed non-linear permeability of the device. This may be due to variations in flow uniformity or differences between parameters such as effective surface area.

It is possible that errors exist in the CFD simulations due to the description of blood as a Newtonian fluid. Newtonian approximations of blood do not include rheological properties such as blood's shear-thinning nature or ability of RBCs to stack at low Re .¹⁴² Yet, with increasing shear rates ($>100 \text{ s}^{-1}$), blood behaves more consistently with Newtonian fluid characteristics due to the dispersion of rouleaux.¹³⁶

Both the bench circuit and the CFD study modeled blood as a Newtonian fluid which may not be representative of blood flow in clinical applications. The aqueous-glycerin solutions

are suitable for Newtonian approximations, but alternative mixtures would display the non-Newtonian, shear-thinning properties of blood. Yet, since the calculated shear rate was greater than 100 s^{-1} for all study flow rates, a Newtonian approximation was assumed. Further, CFD simulations were defined assuming the fluid was experiencing laminar flow according to the calculated Re_i . However, due to the redirection of flow from the inlet tube, the first few layers of the fiber bundle may not be strictly laminar in nature, and more accurate predictions may be achieved from using turbulence models.¹⁴³

Still, the results from this study do not characterize the full measure of fluid velocity, since neither DSA acquisitions nor CFD porous media approximations attempt to resolve the actual fibers. Since, U_t cannot be experimentally determined or simulated within the parameters of this study, the true velocity of the fluid will likely be higher than the physical velocity by some factor, say τ .

Finally, permeability calculations based on acquired pressure measurements may be subject to inaccuracies due to the limitation in accessing pre-fiber pressure drop. It is likely that the significance of this error will be greater at higher flow rates, which may explain the deviation between numerical and experimental pressure drop the 4.5 L/min flow rate.

8. SUMMARY AND CONCLUDING REMARKS

The development of a gas exchange device suitable for long-term respiratory support must balance both biocompatibility and functional design parameters. The dichotomous relationship between factors such as the inflammatory response elicited by a foreign surface versus the performance capacity of the device, and shear-induced platelet activation/cell damage versus low-shear and stagnant areas prone to thrombosis, summarize the delicate balance among a multitude of competing factors. Numerical techniques allow efficient optimization of these variables, and validated CFD analyses offer detailed information to complement clinical and experimental understanding.

The accuracy of CFD models is ultimately limited by the strength of the underlying approximations and assumptions incorporated into the governing equations of fluid flow. Although many design strategies have been tested over the years, most studies confer accuracy based on comparison of numerical and experimental pressure distributions alone, since direct observation of the perfusion dynamics is difficult to accomplish in opaque systems. In fact, to this author's knowledge, the numerical velocity field has never been validated in membranous artificial organs, including kidney hemodialyzers.

Using a clinical X-ray DSA system, details of flow perfusion inside a membrane artificial lung provided quantitative measurements useful for verification of the numerical velocity field. Based on the results of this research, it was shown that when using conventional methods for modeling blood flow through the fiber membrane, despite satisfactory assessments of numerical pressure losses, velocity predictions were significantly lower than experimentally measured. Following a direct comparison between the numerical and experimental values, the incorporation of the resulting scaling constant significantly improved the CFD results.

The validity of the porous media approach remains qualitative in the context of the problem considered in this work; although no other practical approach currently exists. Results of this study suggest the need for experimental validation of flow characteristics which may play a significant role in mass transfer calculations in fiber bundles, assessment of local shear stresses, or designs of the housing geometry. In particular, velocity fields are needed in addition to pressure distributions. Future efforts will need to address improvements to the computational models, but with the aid of X-ray imaging, internal flow dynamics can be evaluated real-time, non-invasively, and with good spatial resolution. The methods developed here will provide researchers tools for more accurate assessments of fluid flow through membrane oxygenators and a clearer understanding of device performance in new prototype designs.

APPENDIX A: NOTATION

a, b	coefficients, —
A	surface area, m^2
A	cross-sectional area, m^2
A_v	wetted surface area, m^2
c	velocity magnitude, pixels/s
C	MCC cross-correlation matrix
d_o	fiber outer diameter, m
d_p	equivalent spherical diameter, m
d_p'	experimental equivalent spherical diameter, m
D	contrast density
\mathcal{D}	diffusivity coefficient, m^2/s
e	dominant error from finite numerical approximations, —
$EtCO_2$	end-tidal CO_2 , mmHg
f, f, f^*	grid function, —
F_B	N.–S. momentum body force term
h	step size (grid spacing)
i	summation or discretization index, $i = (1, 2, \dots, n)$
i	principle direction, $i = (1, 2, 3)^T$
I	MCC search window
\bar{I}	average of the MCC search window
k	permeability, m^2
L	fiber thickness, m
L_e	mean effective length, m
m	index, $m = (1, 2, \dots, N)$
M	array size, pixels
n, N	upper limit
p	pressure, kg/m s
p^*	pressure, —

P	barometric pressure, mmHg
P_{O_2}	oxygen partial pressure, mmHg
q_m	order of the discretization method
Q	flow rate, L/min
Q_b	blood flow rate, L/min
Q_g	sweep gas flow rate, mL/min
r	grid refinement ratio, —
r_h	hydraulic radius, m
Re	Reynolds number, —
Re_i	interstitial Reynolds number, —
S	N.–S. momentum source term
S_{O_2}	oxyhemoglobin saturation, mmHg
t	time, s
t_R	average residence time, s
t_R'	peak-to-peak residence time, s
tHb	hemoglobin concentration, g/dL
T	MCC template
\bar{T}	average template value
T_I	peak intensity of entrance flow, s
T_N	peak intensity of exit flow, s
u, \mathbf{u}	velocity, m/s
u^*	velocity, —
u_{max}	maximum velocity magnitude, m/s
U	superficial velocity, m/s
U'	experimental superficial velocity, m/s
U_{inlet}	inlet superficial velocity, m/s
\bar{U}_p	average physical velocity, m/s
\bar{U}_p'	average physical velocity from time-density analysis, m/s
\bar{U}_p''	average physical velocity imposing a uniform distribution, m/s

U_t	tortuous velocity, m/s
v	distance, pixels
\dot{V}	rate of gas transfer, mL/min
\dot{V}_{CO_2}	rate of CO ₂ transfer, mL/min
\dot{V}_{O_2}	rate of O ₂ transfer, mL/min
V_s	solid volume fraction, m ³
V_t	total volume, m ³
V_v	void volume fraction, m ³
w	distance, pixels
x	distance, m
x, y, z	Cartesian coordinates
<i>Greek Symbols</i>	
α	coefficient of viscous energy losses, —
α_{O_2}	oxygen solubility coefficient in blood, mL/dL/mmHg
β	coefficient of kinetic energy losses, —
$\dot{\gamma}$	shear rate, s ⁻¹
ε	void space (porosity), —
ε_{eff}	effective void space (porosity), —
θ	angle
μ	dynamic viscosity, kg/m s
λ_{O_2}	hemoglobin binding capacity to oxygen, mL/g
ρ	density, kg/m ³
τ	hydraulic tortuosity, —
$\bar{\tau}$	root-mean-squared average shear stress, dyne/cm ²
τ_m	truncation error, $m = (1, 2, \dots, N)$
ω	empirical correction factor, —

APPENDIX B: ABBREVIATIONS

Mathematical operators, common units and variables, and infrequently used molecules or compounds have not been included.

ARDS	Acute Respiratory Distress Syndrome
CAD	Computer Aided Design
CFD	Computational Fluid Dynamics
COPD	Chronic Obstructive Pulmonary Disease
CO ₂	Carbon Dioxide
CPB	Cardiopulmonary Bypass
CT	Computed Tomography
CTA	Constant Temperature Anemometry
DICOM	Digital Imaging and Communications in Medicine
DSA	Digital Subtraction Angiography
ECMO	Extracorporeal Membrane Oxygenation
EM	Electromagnetic (radiation)
FLASH	Fast Low Angle SHot (MRI)
FOV	Field of View
fps	frames per second
Hb	Hemoglobin
MCC	Maximum Cross Correlation
MRI	Magnetic Resonance Imaging
N.–S.	Navier–Stokes
O ₂	Oxygen
OD	Outside (or Outer) Diameter
PC	Phase Contrast
PDE	Partial Differential Equation
PE	Photoelectric (effect)
PIV	Particle Image Velocimetry

PMP	Polymethylpentene
<i>pps</i>	pixels per side
PTV	Particle Tracking Velocimetry
RBC	Red Blood Cell
ROI	Region of Interest
SNR	Signal to Noise Ratio

APPENDIX C: PSEUDO-LANGUAGE ALGORITHMS

<i>Pseudo-Language Variables</i>	
<i>boxsz</i>	Template box dimensions
<i>corrlim</i>	Cross-correlation threshold limit
<i>errvect</i>	Error vector properties (i.e., wrong direction, greater than local standard deviations, etc.)
<i>frames</i>	Set of projection images
<i>gridsz</i>	Spatial discretization of grid points
<i>imdim</i>	Image dimensions
<i>img(-, a, b)</i>	Projection image (possible designation of specific biplane projection)
<i>N</i>	Loop counter
<i>NCC</i>	Normalized cross-correlation matrix
<i>nodes</i>	Vertices for spline construction
<i>nx</i>	Normal direction of a point (i.e., inward or outward); default is positive (inward)
<i>pcloud</i>	3-D point cloud; includes xyz-coordinates and normal direction
<i>spline</i>	2-D quasi-circular spline in z-plane
<i>splineres</i>	Spline resolution (number of points)
<i>tBox</i>	Template box
<i>thresh</i>	Background gray-level intensity
<i>tWin</i>	Search window
<i>winsz</i>	Search window dimensions
<i>Pseudo-Language Functions and Operations</i>	
<i>%</i>	Comment indicator
<i>:</i>	Range
<i>[]</i>	Omit, or remove value
<i>[·]</i>	Function parentheses
<i>(·)</i>	Contents, or subset of
<i>boundary</i>	Traces the boundary of a defined region (e.g., contrast bolus)
<i>count</i>	Count contents
<i>crop</i>	Remove matrix elements
<i>if, else</i>	Conditional statements
<i>max</i>	Maximum value
<i>rotate</i>	Rotate image
<i>run</i>	Execute, or evaluate
<i>search</i>	Search within domain(s) on which to evaluate a function
<i>smooth</i>	Spatial smoothing (i.e., median filter, hole filling, etc.)
<i>spline2d</i>	Execute 2-D cubic-spline interpolation
<i>stdev</i>	Standard deviation

C.1. Image Pre-Processing Subroutine

1. Input *frames*, *imdim*, *thresh*
2. Begin iteration of *frames*, *N*
3. Image pre-processing

```
img = rotate(img)           % image alignment
img = crop(img)             % remove known geometry

if, img(:, :) > thresh      % binary image conversion
    img(:, :) = 1
else, img(:, :) = 0

img = smooth(img)
```

4. [Enter Functions]
5. Repeat *N*
6. End *N*

C.2. Maximum Cross-Correlation

1. Input *corrlim*, *errvect*, *gridsz*, *boxsz*, *winsz*

2. Run C.1.

3. Calculate cross-correlation for each grid point

```
if, tBox(:, :) = boxsz % no calculations for a uniform
                        template pattern
```

```
else, NCC = search(tBox, tWin) % get max correlation
```

```
if, NCC(:, :) < corrlim, end
```

4. Refine *NCC* vectors

```
if, NCC(u, v) = errvect % sans standard deviation
```

```
NCC(u, v) = []
```

5. Only keep highest cross correlation for each grid point

```
NCC(N) = max(NCC(1:N))
```

6. Conduct filtering of final vector field

```
if, NCC(:, :) = errvect(stdev)
```

```
NCC(u, v) = []
```

7. Convert units: pixel/frame to mm/s

C.3. Point Cloud Reconstruction

1. Input *splineres*
2. Run C.1.
3. Identify outer edge of contrast perfusion

```
imga = boundary(imga)  
imgb = boundary(imgb)  
  
smooth(imga, imgb)           % ensure continuous boundary (i.e., no  
                               holes)  
  
nodes = count(imga, imgb)   % repeat for each row/slice
```

4. Refine nodes

```
if, nodes = 4 ∨ 12           % a slice must contain exactly 4 or 12  
                               nodes; at least 2 from each img  
  
    if, [nodes = 4] ∧ [count(imga) = 2]  
        else, nodes = []  
    else, nodes = []
```

5. Create 2-D spline for each slice

```
spline = spline2d(nodes, splineres)
```

6. Export point cloud for volumetric reconstruction

```
pcloud = (nodes, slice, nx)           % contains xyz-coordinates, and  
                                           normal direction  
  
if, nodes = 4  
else,  
    plcloud((2, 5, 8, 11), nx) = -nx   % outward normal  
                                           (relative to inlet tube)
```

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Publications (Peer Reviewed)

1. De Bartolo C, Nigro A, Fragomeni G, Colacino FM, Wang D, **Jones CC**, and Zwischenberger JB. (2011). “Numerical and Experimental Flow Analysis of the Wang-Zwische Double Lumen Cannula.” *ASAIO*, 57(4):318–27.
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1. **Jones CC**, Capasso P, Wang D, McDonough JM, Rosenstein KS, and Zwischenberger JB. “Biplane Angiography for Experimental Validation of CFD Models of Blood Flow in Artificial Lungs.” *Submitted*.
2. Ballard-Croft C, Wang D, **Jones CC**, Zhou X, Sumpter LR, and Zwischenberger JB. “A New, Simplified Systemic Hyperthermia Circuit for Advanced Lung Cancer Treatment.” *Submitted*.

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1. Niermann KJ, Fleischer AC, Jarquin-Valdivia A, **Jones CC**, Lefkovitz E, Donnelly E, and Martin DS. (2004). “Sonographic Assessment of Skeletal Muscles: Implications for a Manned Mission to Mars.” In DF Crossman & DR Zubrin (Eds.), *On To Mars 2—Exploring and Settling a New World* (Vol. 55). Burlington, Ontario: Apogee Books Space Series.

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6. **Jones CC**. (2010). "Design in the Biomedical Industry." *Biomedical Engineering Seminar Series*, Lexington, KY.
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