TECHNICAL TRANSACTIONS CZASOPISMO TECHNICZNE

MECHANICS MECHANIKA

1-M/2013

KAROL OSOWSKI*, MARCIN MIGUS**, ANDRZEJ KĘSY*

EXPERT SYSTEM FOR SUPPORTING THE PROCESS OF HYDRODYNAMIC TORQUE CONVERTER CONSTRUCTION

SYSTEM EKSPERTOWY WSPOMAGAJĄCY PROCES KONSTRUOWANIA PRZEKŁADNI HYDROKINETYCZNEJ

Abstract

This paper presents an advisory expert system to assist the process of constructing the hydrodynamic torque converter. The system has been constructed using program Delphi 7 Enterprise in Object Pascal. The system includes three types of means of transport, a car, a rail bus and a working machine.

Keywords: *expert systems*, *computer aided design*, *hydrodynamic torque converter*

Streszczenie

W artykule przedstawiono doradczy system ekspertowy wspomagający proces konstruowania przekładni hydrokinetycznej. System zbudowano z użyciem programu *Delphi 7 Enterprise* w języku *Object Pascal*. W systemie tym uwzględniono trzy rodzaje środków transportu, w którym przekładnia hydrokinetyczna pracuje: samochód osobowy, autobus szynowy i maszynę roboczą.

Słowa kluczowe: *systemy ekspertowe*, *komputerowe wspomaganie konstrukcji*, *przekładnia hydrokinetyczna*

^{*} MSc. Karol Osowski, prof. PhD. Eng. Andrzej Kęsy Institute of Applied Mechanics and Energetics, Faculty of Mechanical Engineering , Kazimierz Pulaski University of Technology and Humanities in Radom.

^{**} PhD. Eng. Marcin Migus, The State School of Higher Education in Sandomierz.

Symbols

- η_{75} economic range of operation
- η^* maximum efficiency
- χ_p computational durability
 λ_{M} moment coefficient
- l*^M*,*ⁱ* moment coefficient
- α_1 , α_2 damping factor for pump and turbine impellers
- *S* execution cost
- *g* metal consumption factor
- i_{ω} *^d*⁰ – dynamic transmission ratio
- n_p idle gear loss coefficient
- $i_{\scriptscriptstyle M}$ i_M – coefficient of torque limit
 λ_{M} – coefficient of axial force
- l*^A*,*ⁱ* coefficient of axial force
- *i* – coefficient of speed reduction
- *p* permeability
- *X* coefficient of stiffness performance
- β_{11} blade angle at the entrance of the pump impeller
 β_{12} blade angle at the exit of the pump impeller
-
- β_{12} blade angle at the exit of the pump impeller
 β_{21} blade angle at the entrance of the turbine im β_{21} – blade angle at the entrance of the turbine impeller
 β_{22} – blade angle at the exit of the turbine impeller
- blade angle at the exit of the turbine impeller
- β_{31} blade angle at the entrance of the stator impeller
 β_{32} blade angle at the exit of the stator impeller
-
- β_{32} blade angle at the exit of the stator impeller r_{12} radius of the average line at the exit to the p - radius of the average line at the exit to the pump impeller
- r_{21} radius of the average line at the entrance to the turbine impeller
- r_{22} radius of the average line at the exit to the turbine impeller
- J_1, J_2 moments of inertia of the input shaft
- c_1, c_2 stiffness of the input shaft
- $l₂$ – length of the average line

1. Introduction

A hydrodynamic transmission system (HTS) is a part of the propulsion system in the ground transportation using the kinetic energy of the working fluid. No permanent connection between the input shaft and the output shaft causes no shock loads transferred from the working tool to the propulsion system. The main advantage of the HTS is shifting in relation to the applied load. Furthermore, it effectively reduces the dynamic load and damping vibration, thus highly increasing the durability of the propulsion system [1]. One of the major disadvantages of the HTS is low maximum efficiency and narrow range of high efficiency as a function of the speed ratio. The basic element of a HTS consists of an hydrodynamic sub-assembly which includes: torque converters, clutch or brake. The most complicated component in the design is the hydrodynamic torque converter (HTC) which is composed of three impellers: pump, turbine and the stator as it is shown in Figure 1.

Fig. 1. Construction of the hydrodynamic torque converter [2]: *P* – pump impeller, *T* – turbine impeller, *K* – stator impeller Rys. 1. Budowa przekładni hydrokinetycznej [2]: *P* – koło łopatkowe pompy, *T* – koło łopatkowe

turbiny, *K* – koło łopatkowe kierownicy

The impellers consist of blades that with together the housing create the work area for the working fluid.

The introduction of modern low-speed engines using alternative fuels, the requirement of exhaust emissions reduction and the increase of the efficiency of the propulsion system reducing fuel consumption means that the torque converter design process is very current and important. Too low accuracy of existing mathematical models results in a large discrepancy between the requirements and the resulting characteristics of the HTC. The failure to comply with the prescribed requirements results in the deterioration of the characteristics of the propulsion system and forms of economic losses.

The development of the methodology supporting the process of constructing the HTC by using expert system can resolve the existing problems of the construction. The benefits of using this solution can include not only economic aspects in the form of increasing the efficiency, but also the optimal choice of torque converter for a given mode of transport.

Expert systems are complex computer programs that use knowledge derived from subject matter experts sources. They consist of three main components: the interface, inference engine and a knowledge base. Such systems are increasingly being used in the design and operation of machines and equipment. They can be divided into advisory, diagnostic and decision-making systems. Are widely used in supporting the construction process advisory systems [3], their main task is to solve a problem using the knowledge contained within the database.

This paper presents an advisory system for supporting the process of design the HTC.

2. Advisory expert system to assist in the process of designing a hydrodynamic torque converter

The presented system in the paper the which support the design of a HTC is an advisory expert system software based on the collected data [4]. The system has been developed using *Delphi 7 Enerprise* software with the help of object-oriented programming language ‒ *Object Pascal*. The result of the working system is the choice of the optimum design solution for the design of a HTC. Solving the decision-making problems in the system is done by selecting:

- optimal torque converter evaluation index for the predefined means of transport,
- optimal design parameters having an impact on the rate of the highest evaluation,
- field variation of the optimal design parameters.

The diagram of the system in the form of a decision tree is shown in Figure 2.

Fig. 2. A decision tree system supporting the process of designing a hydrodynamic torque converter Rys. 2. Drzewo decyzyjne systemu wspomagającego proces konstruowania przekładni hydrokinetycznej

The selection of the optimal values of the assessment indicators, construction parameters and their ranges of variation have been implemented for a passenger car, a rail bus and a working machine.

A knowledge base of the advisory expert system to support the process of designing the hydrodynamic torque converter has been developed on the basis of domain subject matter experts opinions and professional literature in the field of HTS such as: scholarly articles, books, standards, patents. The knowledge base includes:

- assessment indicator impact on a means of transport,
- the impact of construction parameters on the assessment indicators,
- ranges of the construction parameters variability.

The knowledge base has been divided into various sources of data provenance. The data obtained during the point assessment method as well as the data from books, articles, JCR and standards [5] has been implemented. The data coming from various literature sources has been divided into experimental and theoretical ones. In addition, arbitrary coefficients of confidence have been introduced for all the data. A division of the knowledge base fragment is shown in Figure 3.

```
SYEXP2
(MASZYNA ROBOCZA)
  Procedure Danelm; (Dane dla etamax - maszyna robocza)
 begin
  ocenaeksperta:= 161; maxpunktow:=180;
  wksiazkach: = 3; wartykulachJCR: = 7; winnych: = 9;
  eksperymentalne:=10; teoretyczne:=9;
  u1:=0.9; u2:=0.9; u3:=0.9;end:
  Procedure Dane2m; (Dane dla id0max - maszyna robocza)
 begin
  ocenaeksperta:= 178; maxpunktow:=180;
  wksiazkach: = 4; wartykulachJCR: = 6; winnych: = 2;
  eksperymentalne:= 10; teoretyczne:=2;
  u1:=1; u2:=1; u3:=0.9;
 end:
```
Fig. 3. A division of the knowledge base fragment

Rys. 3. Podział fragmentu bazy wiedzy

The inference engine retrieves data from a knowledge base with some weights. The value of "1" is assigned to the data coming from the books, the value of "0.8" to the articles of the JCR and the value of "0.5" to other literature sources.

3. Indicators assessment and design parameters

Completely different working conditions of ground transportation cause that the propulsion system is subjected to different types of resistance, and hence there are various types of loads. The HTC design requires the determination of the requirements for a given means of transport, including the expected load, or cooperation with the propulsion system. To ensure the optimal design the following requirements kinematic, dynamic, productive, economical and exploitation ones should be taken into account in order to ensure the optimal design. Assessment indicators have been introduced to check if HTC meets the requirements and to assess the impact of design parameters on the design process [2, 5]. When selecting a HTC for a given means of transport which operates in very specific conditions, the assessment indicator is to achieve its optimal value.

Assessment indicators chosen for the HTC which have been subjected to the expert point assessment method [5–7] have been selected they are affected by a large number of construction parameters: η_{75} , η^* , χ_p , $\lambda_{M,i}$, α_1 , α_2 , *S*, *g*, *i_{d0}*, n_p , *i_{d0}*, $\lambda_{A,i}$, *i_g*, *p*, *X*. Next the indicators have been evaluated by subject – matter experts. The level of validity of the assessment indicators for a given means of transport using the point assessment method [5] has been evaluated by experts. The indicators have been assessed on the scale 0 to 180. The results are shown in Table 1.

For each means of transport three assessment indicators with most points have been selected and they have been implemented to the knowledge base of the advisory expert system.

In order to obtain optimum values of the HTC assessment indicators is the structural parameters directly affecting the assessment indicators must be changed. Design parameters with greatest impact on the assessment indicators evaluated by means of the assessed using point assessment expert method [4, 6], have been selected. They are: β_{22} , β_{32} , β_{12} , r_{12} , J_1 , r_{21} , $r_{22}, J_2, \beta_{21}, \beta_{11}, c_1, c_2, \beta_{31}, l_2.$

Table 1

The parameters were subjected to subject matter experts evaluation. The experts assessed the impact of various design parameters on the indicators [5] by using the point assessment method. The parameters were assessed on 0 to 180 points scale. The results are shown in Table 2.

Three design parameters with the highest score have been selected and implemented to the knowledge base of the advisory expert system.

The ranges of design indicator variability have been selected on the base of current HTC construction and they have been evaluated by subject matter experts on 0 to 180 scale. The results are shown in Table 3.

Table 2

Design parameters impact on the indicators of the hydrodynamic torque converter

η^* i_{d0} η_{75} \boldsymbol{p} 118 168 162 128 β_{22}	
122 172 143 166 β_{32}	
162 149 β_{12} 140 136	
112 98 85 75 r_{12}	
48 ${\cal J}_1$ $88\,$ 98 102	
66 $78\,$ 66 59 r_{21}	
88 104 67 100 r_{22}	
62 ${\cal J}_2$ 74 84 60	
β_{21} 166 122 116 162	
141 144 β_{11} 168 138	
110 66 112 $78\,$ c_{1}	
110 88 94 90 c_{2}	
175 128 144 170 β_{31}	
65 l_{2} 66 70 50	

Table 3

Ranges of design of parameter variability

	Range I	Range II	Range III		Range I	Range II	Range III
β_{22}	129–139	139-148	$148 - 160$	β_{21}	$30 - 45$	$45 - 50$	$50 - 75$
Rating	122	164	125	Rating	149	88	138
β_{32}	$20 - 30$	$30 - 40$	$40 - 50$	β_{11}	$90 - 102$	$102 - 114$	$114 - 130$
Rating	158	98	48	Rating	111	166	102
β_{12}	$75 - 100$	$100 - 125$	$125 - 150$	β_{31}	$70 - 92$	$92 - 114$	$114 - 140$
Rating	74	148	134	Rating	66	165	78

4. The result of the system performance

The optimal design solution suggested by the described expert system for the HTC is following:

- for a passenger car assessment index *p* and the construction parameter β_{12} (75° 100°),
- for rail bus assessment index i_{d0} and the construction parameter β_{12} (125^o 150^o),
- for a working machine evaluation index η* and the construction parameter β_{32} (20° 30°).

5. Conclusions

- 1. The expert system for each means of transport chooses the assessment indicator which is to achieve the max. The process is carried out by choosing the design parameter and the variability range of this parameter.
- 2. Inference system is based on the data contained in the knowledge base.
- 3. The knowledge base is separated from the rest of the program, which allows for continuous modification and the data completion.

References

- [1] Kęsy A., Kęsy Z., *Damping Characteristics of a Transmission System with a Hydrodynamic Torque Converter*, J. of Sound and Vibration, 16/3, 1993.
- [2] Kęsy A., *Modele bryłowe w konstrukcji podzespołów hydrokinetycznych*, Wydawnictwo Politechniki Radomskiej, Radom 2012.
- [3] Pokojski J., *Systemy doradcze w projektowaniu maszyn*, WNT, Warszawa 2005.
- [4] Osowski K., Migus M., Kęsy A., *Assumptions for Expert System Creation to Assess Durability of Hydrodynamic Transmission System*, The XIIth International Scientific IFToMM Conference "Tribology and Reliability", Sankt Petersburg 24–26 October 2012, CD, 2012.
- [5] Stesin S.P., *Optymalizacja parametrów hydrodynamicznych przewodów maszyn budowlanych i drogowych*, *Budowa maszyn,* Moskwa 1996 (tłum. autor z cyrylicy).
- [6] Kęsy A., *Numeryczna identyfikacja i optymalizacja napędów hydrokinetycznych środków transport*u, Wydawnictwo Politechniki Radomskiej, Radom 2004.
- [7] Kęsy Z., Kęsy A., Madeja J., *Identyfication of Hydrodynamic Torque Converter Controlled by Physical Properties of Working Fluid*, Int. Conference "Modern Practice In Stress and Vibration Analysis", Dublin 1997.