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DIGITAL MODEL PREDICTIVE CONTROL OF THE THREE TANK SYSTEM BASED ON LAGUERRE FUNCTIONS*

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Abstract. *The application of the model predictive control (MPC) based on discrete-time Laguerre functions is presented in this paper. A nonlinear three-tank hydraulic system is used as an object to which the proposed algorithm is applied. The paper also presents the method of linearization of the nonlinear system, as well as the procedure for the controller design. For the verification of the proposed control method, digital simulations are performed using Matlab.*

Key words: *model predictive control, hydraulic system, linearization, Laguerre functions*

1. INTRODUCTION

Model Predictive Control (MPC) is a control method that has many applications in the different fields of engineering. For a long time, it has been in the focus of interest of both engineering and academic circles. In a sense, it represents a further development of optimal control algorithms for linear systems developed in the 50s and 60s of the last century [1], [2]. MPC has traditionally been used to control a system with relatively slow dynamics, but recently, this method for the linear and nonlinear systems control has found application in the chemical, petroleum and other industries [3], [4], due to more efficient optimization formulations and the availability of computational power.

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The main idea is to use a dynamical model and an optimization formulation to optimize the predicted future plant behaviour, with future inputs as degrees of freedom in the optimization [5]. The optimization is performed for a finite prediction horizon into the future but re-optimized at every time step (hence the alternative name Receding Horizon Control). While the optimal control action over a future time horizon is calculated, only the first control action is implemented before the calculations are repeated at the next sample instant. This feature distinguishes MPC from other optimal control algorithms whose control law is determined off-line, i.e. it is pre-calculated.

For the application of a digital MPC, it is possible to use various models of the system, such as the impulse response model, the transfer function model, the state space model, etc. In this paper, the state space model is used, whose main advantage is an easy representation of the system with multiple inputs and multiple outputs [6].

To cope with the time needed for the calculation of the optimal control move, as well as the number of the parameters used for the fast calculations when the long horizon is used, different methods can be introduced within the traditional MPC algorithm. One of them is using orthogonal functions representations of the predicted control trajectories. This paper shows how Laguerre functions can be applied in this manner [7].

The system of three reservoirs is considered in this paper [8]. The primary control problem for this system is to achieve the desired fluid level in each of the tanks and keep this level constant. The control of the levels in all three tanks is achieved by the pump control, which serves to supply fluid to the system, and by automatic opening and closing valves that provide desired fluid flow.

The paper is organized in the following way. Section 2 provides a detailed description of the system with three reservoirs. Section 3 describes a nonlinear model of the system, and the following, Section 4, shows the process of linearizing the model of the system. Section 5 explains the design and application of the digital MPC based on Laguerre functions, and in Section 6, the simulated results are given. Section 7 contains conclusions and remarks on further work.

2. THE THREE-TANK SYSTEM

The three-tank system consists of three physically separated reservoirs that, at their bottom, have valves whose opening allows the water to flow out of the tank. All three tanks are of different shapes. The first reservoir, viewed from above, has a constant cross-section. The other two reservoirs are spherical and conical, so they have different cross sections, which create basic nonlinearities in the system. To fill the fluid in the first tank, a variable flow pump is used. Fluid, due to gravity, goes into the other two reservoirs, and the flow is regulated by opening and closing the aforementioned controlled valves. There is a fluid level meter in each tank. Below the third tank is an auxiliary tank used to collect the fluid that flows from the third tank so that the same fluid is reused for charging the first tank with the pump. The block diagram of the described system is given in Fig. 1.

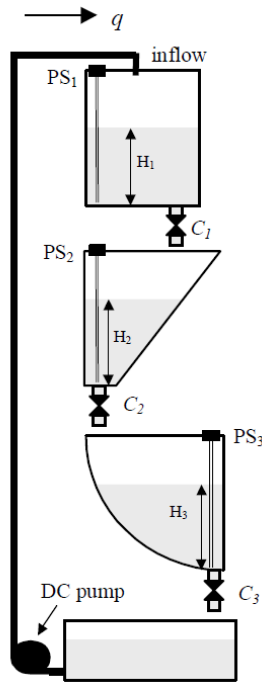


Fig. 1 The three-tank system

It is assumed that the valve-controlled system is considered, and the flow of the fluid, provided by the DC pump, is a constant parameter (Fig. 2).

The states of the system H_1 , H_2 , and H_3 represent the levels of the fluid in the first, second and third reservoirs, respectively.

The system is controlled by three inputs:

- u_1 – the control of the valve C1,
- u_2 – the control of the valve C2, and
- u_3 – the control of the valve C3.

The possibility of occurrence of fluctuations in fluid level control in reservoirs could be produced by nonlinearities due to reservoirs shape, valve geometry and dead zone of the valves.

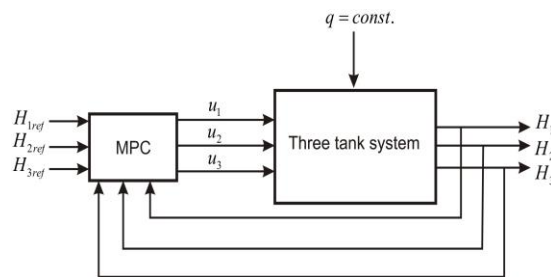


Fig. 2 Valve-controlled three tank system

The laminar outflow rate of an “ideal” fluid can be calculated according to Bernoulli law:

$$Q_r = \mu S \sqrt{2gH_r}, \quad r = \overline{1,3}, \quad (1)$$

where μ is the orifice outflow coefficient, S is the output area of the orifice, H_r is the level of the liquid in the tank r , and g is the gravitational acceleration.

If the value for the openness of the valve is defined by $S = u_r S_{0r}$, $0 \leq u_r \leq 1$, where S_{0r} is fully opened valve constant, u_r is the coefficient which determines how much the valve is opened, the following equation is obtained:

$$Q_r = \mu u_r S_{0r} \sqrt{2gH_r} = D_r u_r H_r^{\alpha_r}, \quad r = \overline{1,3}, \quad (2)$$

where $\alpha_r = 1/2$, $D_r = \mu S_{0r} \sqrt{2g}$. α_r is the value which depends on turbulence and acceleration of the liquid, and it can be different for the different kind of fluids.

The system dynamics is described by:

$$\begin{aligned} \frac{dV_1}{dH_1} \frac{dH_1}{dt} &= q - D_1 u_1 H_1^{\alpha_1}, \\ \frac{dV_2}{dH_2} \frac{dH_2}{dt} &= D_1 u_1 H_1^{\alpha_1} - D_2 u_2 H_2^{\alpha_2}, \\ \frac{dV_3}{dH_3} \frac{dH_3}{dt} &= D_2 u_2 H_2^{\alpha_2} - D_3 u_3 H_3^{\alpha_3}. \end{aligned} \quad (3)$$

From (3), it can be obtained:

$$\begin{aligned} \frac{dV_1}{dH_1} &= \beta_1(H_1) = aw, \\ \frac{dV_2}{dH_2} &= \beta_2(H_2) = cw + \frac{H_2}{H_{2\max}} bw, \\ \frac{dV_3}{dH_3} &= \beta_3(H_3) = w \sqrt{R^2 - (R - H_3)^2}, \end{aligned} \quad (4)$$

where $\beta_i(H_i)$ is a cross sectional area of the i -th tank at the level H_i , and the other parameters of the tanks are (Fig. 3): $a = 0.25$ m, $w = 0.035$ m, $c = 0.1$ m, $b = 0.345$ m, $R = 0.364$ m, $H_{1\max} = H_{2\max} = H_{3\max} = 0.35$ m.

Substituting (4) in (3) yields:

$$\begin{aligned}
 \frac{dH_1}{dt} &= \frac{1}{\beta_1(H_1)} q - \frac{1}{\beta_1(H_1)} D_1 u_1 H_1^{\alpha_1}, \\
 \frac{dH_2}{dt} &= \frac{1}{\beta_2(H_2)} D_1 u_1 H_1^{\alpha_1} - \frac{1}{\beta_2(H_2)} D_2 u_2 H_2^{\alpha_2}, \\
 \frac{dH_3}{dt} &= \frac{1}{\beta_3(H_3)} D_2 u_2 H_2^{\alpha_2} - \frac{1}{\beta_3(H_3)} D_3 u_3 H_3^{\alpha_3}.
 \end{aligned} \tag{5}$$

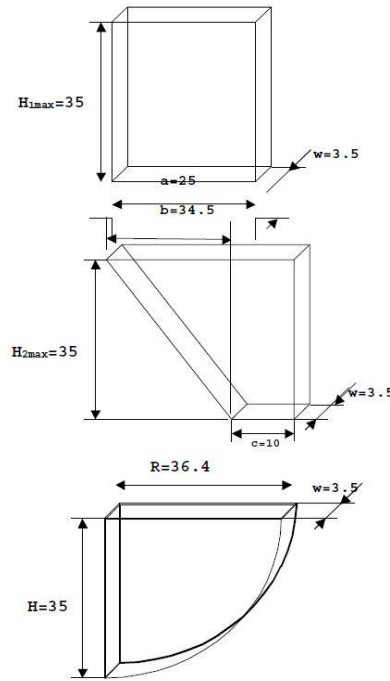


Fig. 3 Dimensions of the tanks

The last set of equations is a nonlinear model of the observed system and completely describes its dynamics. In order to implement the proposed control algorithm, a linearized model of this system is required.

3. LINEARIZED MODEL OF THE THREE TANK SYSTEM

The equilibrium point of the system is obtained by equating the right side of the (5) with zero:

$$\begin{aligned}
 0 &= q - D_1 u_{10} H_{10}^{\alpha_1}, \\
 0 &= D_1 u_{10} H_{10}^{\alpha_1} - D_2 u_{20} H_{20}^{\alpha_2}, \\
 0 &= D_2 u_{20} H_{20}^{\alpha_2} - D_3 u_{30} H_{30}^{\alpha_3}.
 \end{aligned} \tag{6}$$

Using (6), the equilibrium point is calculated by:

$$q_0 = D_1 u_{10} H_{10}^{\alpha_1} = D_2 u_{20} H_{20}^{\alpha_2} = D_3 u_{30} H_{30}^{\alpha_3}. \quad (7)$$

By denoting

$$dH_i/dt = F_i(q, H_i, u_i),$$

the equation (5) can be reformulated as:

$$\begin{aligned} F_1(q, H_1, u_1) &= \frac{q}{\beta_1(H_1)} - \frac{D_1 u_1 H_1^{\alpha_1}}{\beta_1(H_1)}, \\ F_2(H_1, H_2, u_1, u_2) &= \frac{D_1 u_1 H_1^{\alpha_1}}{\beta_2(H_2)} - \frac{D_2 u_2 H_2^{\alpha_2}}{\beta_2(H_2)}, \\ F_3(H_2, H_3, u_2, u_3) &= \frac{D_2 u_2 H_2^{\alpha_2}}{\beta_3(H_3)} - \frac{D_3 u_3 H_3^{\alpha_3}}{\beta_3(H_3)}. \end{aligned} \quad (8)$$

Using the Taylor expansion of (8), around the obtained equilibrium point, the linearized model is obtained by

$$\frac{dh}{dt} = J_H h + J_u u, \quad (9)$$

where $h = H - H_0$ is a deviation from the equilibrium point H_0 , $u = u - u_0$ is a deviation from the control u_0 . Jacobian matrices of the (8) are then determined by

$$J_h = \left[\frac{\partial F_i(H_i, q_i, u_i)}{\partial H} \right], J_u = \left[\frac{\partial F_i(H_i, q_i, u_i)}{\partial u} \right] \text{ for } H_i = H_{i0}, q = q_0, u_i = u_{i0}.$$

For the three tank system described by (8), the obtained Jacobians have the following form:

$$J_h = \begin{bmatrix} J_{h11} & J_{h12} & J_{h13} \\ J_{h21} & J_{h22} & J_{h23} \\ J_{h31} & J_{h32} & J_{h33} \end{bmatrix}, J_u = \begin{bmatrix} J_{u11} & J_{u12} & J_{u13} \\ J_{u21} & J_{u22} & J_{u23} \\ J_{u31} & J_{u32} & J_{u33} \end{bmatrix}$$

$$\begin{aligned} \text{where } J_{h11} &= -\frac{D_1 u_{10} \alpha_1 H_{10}^{\alpha_1 - 1}}{\beta_1(H_{10})}, J_{h12} = 0, J_{h13} = 0, J_{h21} = \frac{D_1 u_{10} \alpha_1 H_{10}^{\alpha_1 - 1}}{\beta_2(H_{20})}, \\ J_{h22} &= -\frac{D_2 u_{20} \alpha_2 H_{20}^{\alpha_2 - 1}}{\beta_2(H_{20})}, J_{h23} = 0, J_{h31} = 0, J_{h32} = \frac{D_2 u_{20} \alpha_2 H_{20}^{\alpha_2 - 1}}{\beta_3(H_{30})}, \\ J_{h33} &= -\frac{D_3 u_{30} \alpha_3 H_{30}^{\alpha_3 - 1}}{\beta_3(H_{30})}, J_{u11} = -\frac{D_1 H_{10}^{\alpha_1}}{\beta_1(H_{10})}, J_{u12} = 0, J_{u13} = 0, J_{u21} = \frac{D_1 H_{10}^{\alpha_1}}{\beta_2(H_{20})}, \\ J_{u22} &= -\frac{D_2 H_{20}^{\alpha_2}}{\beta_2(H_{20})}, J_{u23} = 0, J_{u31} = 0, J_{u32} = \frac{D_2 H_{20}^{\alpha_2}}{\beta_3(H_{30})}, J_{u33} = -\frac{D_3 H_{30}^{\alpha_3}}{\beta_3(H_{30})}. \end{aligned}$$

Now, the linearized model of the system can be represented in the form of a linear differential equation:

$$\dot{h}(t) = Ah + Bu, \quad (10)$$

where matrices A and B are already obtained Jacobians J_h and J_u , respectively.

4. DESIGN OF MPC OF THREE TANK SYSTEM

In order to define the increment of the control signal $\Delta \mathbf{u}$, and to design MPC based on Laguerre functions, the augmented model of the system is obtained by introducing the integrators:

$$\begin{bmatrix} \Delta \mathbf{x}_m(k+1) \\ \mathbf{y}(k+1) \end{bmatrix} = \begin{bmatrix} \mathbf{A}_m & \mathbf{0}_m^T \\ \mathbf{C}_m \mathbf{A}_m & \mathbf{I}_{q \times q} \end{bmatrix} \begin{bmatrix} \Delta \mathbf{x}(k) \\ \mathbf{y}(k) \end{bmatrix} + \begin{bmatrix} \mathbf{B}_m \\ \mathbf{C}_m \mathbf{B}_m \end{bmatrix} \Delta \mathbf{u}(k), \quad (11)$$

$$\mathbf{y}(k) = \begin{bmatrix} \mathbf{0}_m & \mathbf{I}_{q \times q} \end{bmatrix}.$$

where matrices \mathbf{A}_m , \mathbf{B}_m , and \mathbf{C}_m are discrete-time matrices of the system.

The optimized control trajectory

$$\Delta \mathbf{u}(k) = [\Delta u_1(k) \quad \Delta u_2(k) \quad \dots \quad \Delta u_{N_c}(k)], \quad (12)$$

consists of the increments of the control signal in the control horizon N_c and it is calculated by

$$\Delta \mathbf{u}_i(k) = \mathbf{L}_i(k)^T \boldsymbol{\eta}_i \quad (13)$$

where

$$\mathbf{L}_i(k)^T = [l_1^i(k) \quad l_2^i(k) \quad \dots \quad l_N^i(k)], \quad (14)$$

is Laguerre functions vector, and $\boldsymbol{\eta}_i$ is Laguerre coefficient vector [7]. The parameters of the Laguerre functions are determined from the difference equation of the discrete-time Laguerre functions defined as:

$$\mathbf{L}_i(k+1) = \mathbf{A}l_i \mathbf{L}_i(k), \quad (15)$$

where the matrix $\mathbf{A}l_i$, containing the parameters of the Laguerre network, φ and $\gamma = 1 - \varphi^2$, is given in the following form:

$$\mathbf{A}l_i = \begin{bmatrix} \varphi & 0 & 0 & \dots & 0 \\ \gamma & \varphi & 0 & \dots & 0 \\ -\varphi\gamma & \gamma & \varphi & \dots & 0 \\ -\varphi^{N-2}\gamma & -\varphi^{N-3}\gamma & \dots & -\varphi^{N-N}\gamma & \varphi \end{bmatrix} \quad (16)$$

and the vector of the initial conditions is

$$\mathbf{L}_i(0) = \sqrt{\gamma} [1 \quad -\varphi \quad \varphi^2 \quad -\varphi^3 \quad \dots \quad (-1)^{N-1} \varphi^{N-1}]. \quad (17)$$

The optimal control action is then calculated by

$$\Delta \mathbf{u}(k_i) = \begin{bmatrix} \mathbf{L}_1(0)^T & \mathbf{0}_2^T & \cdots & \mathbf{0}_m^T \\ \mathbf{0}_1^T & \mathbf{L}_2(0)^T & \cdots & \mathbf{0}_m^T \\ & & \ddots & \\ \mathbf{0}_1^T & \mathbf{0}_2^T & \cdots & \mathbf{L}_m(0)^T \end{bmatrix} \boldsymbol{\eta}_i \quad (18)$$

where m represents the number of inputs.

5. DIGITAL SIMULATION RESULTS

After the linearization method described in Section 3, the following Jacobian matrices of the three tank system are obtained:

$$\mathbf{J}_h = \begin{bmatrix} -0.0274 & 0 & 0 \\ 0.0344 & -0.0366 & 0 \\ 0 & 0.0290 & -0.0297 \end{bmatrix}, \quad (19)$$

$$\mathbf{J}_u = \begin{bmatrix} -0.0041 & 0 & 0 \\ 0.0052 & -0.0052 & 0 \\ 0 & 0.0041 & -0.0042 \end{bmatrix}.$$

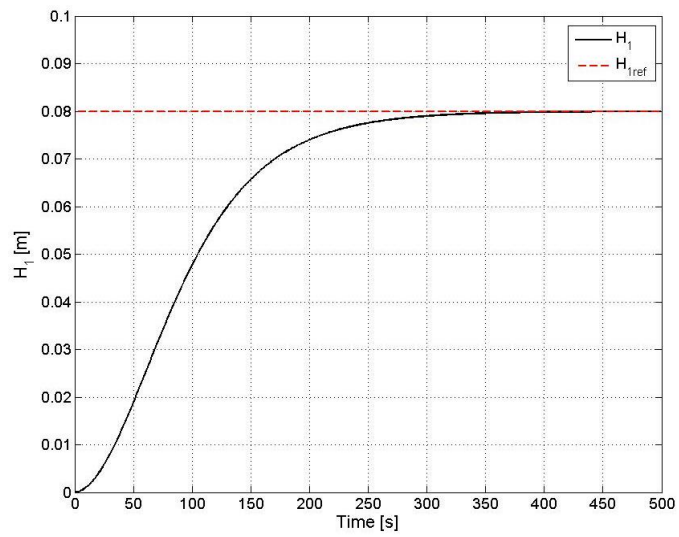
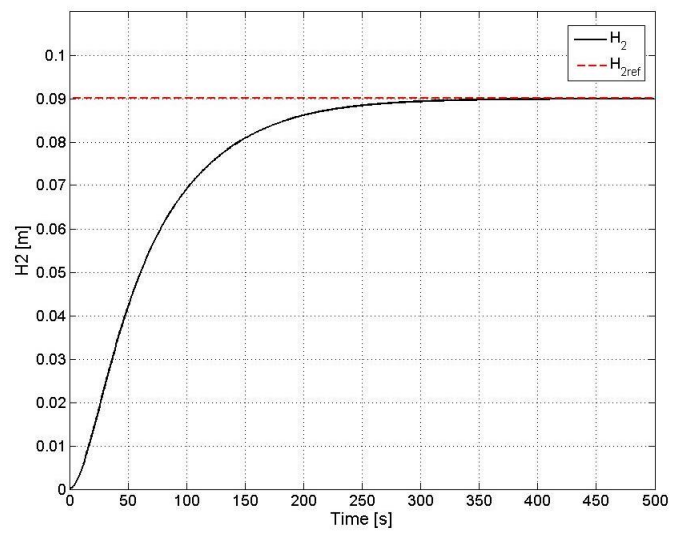
The parameters that are used for the Jacobians and the simulations are:

- $H_{10} = 0.1\text{m}, H_{20} = 0.1\text{m}, H_{30} = 0.1\text{m},$
- $q_0 = 2.7958e - 005 \text{ m}^3 / \text{s},$
- $u_{10} = 0.7733, u_{20} = 0.7734, u_{30} = 0.7676,$
- $D_1 = 7.3322e - 005, D_2 = 7.667e - 005, D_3 = 7.8632e - 005,$
- $\alpha_1 = 0.3071, \alpha_2 = 0.3265, \alpha_3 = 0.3342.$

The simulations are conducted, and the robustness of the proposed controller is examined. As it was mentioned before, the equilibrium point is calculated for the equal value of the tanks levels of $H_{10} = H_{20} = H_{30} = 0.1\text{m}$, and the referent tank level values are chosen as:

$$H_{1\text{ref}} = 0.08\text{m}, H_{2\text{ref}} = 0.09\text{m}, H_{3\text{ref}} = 0.11\text{m}.$$

It can be seen that the referent levels are achieved using the proposed MPC, and the desired values of the fluid levels for the first, second and third tank are shown in Figs. 4, 5 and 6, respectively:

**Fig. 4** Level H_1 **Fig. 5** Level H_2

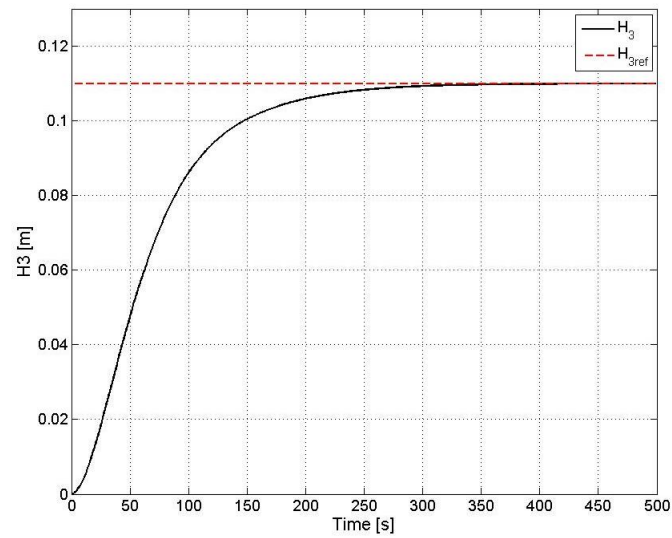


Fig. 6 Level H_3

The proposed discrete-time MPC based on the Laguerre functions, defined by (17), applied to the plant described by (10) results in the control signals depicted in Figs. 7, 8 and 9.

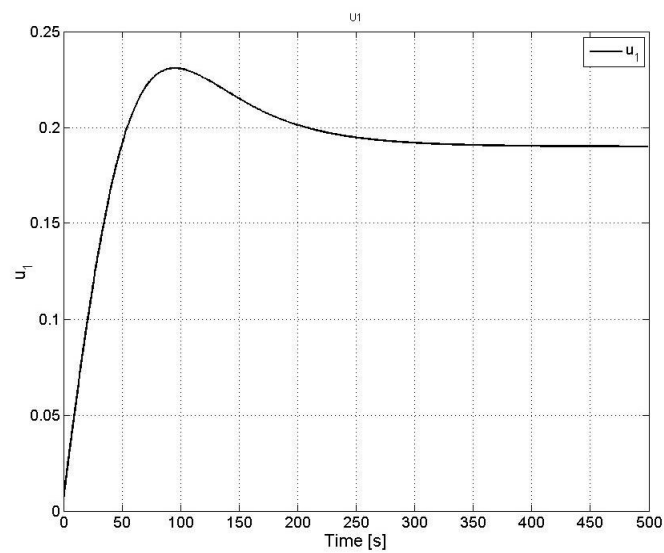
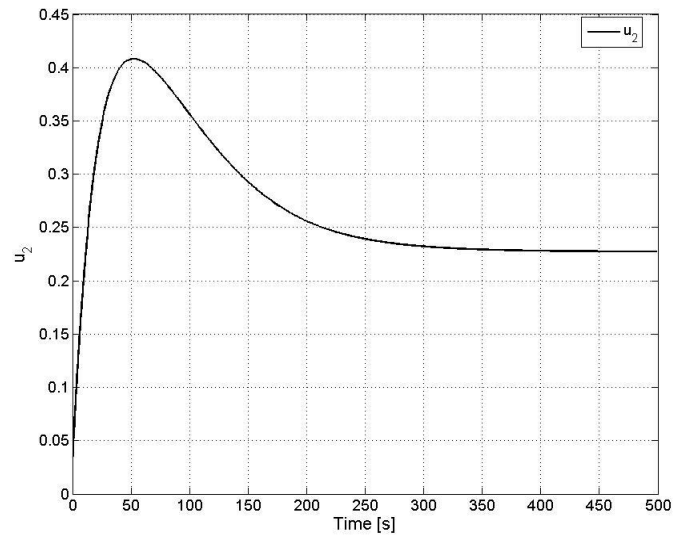
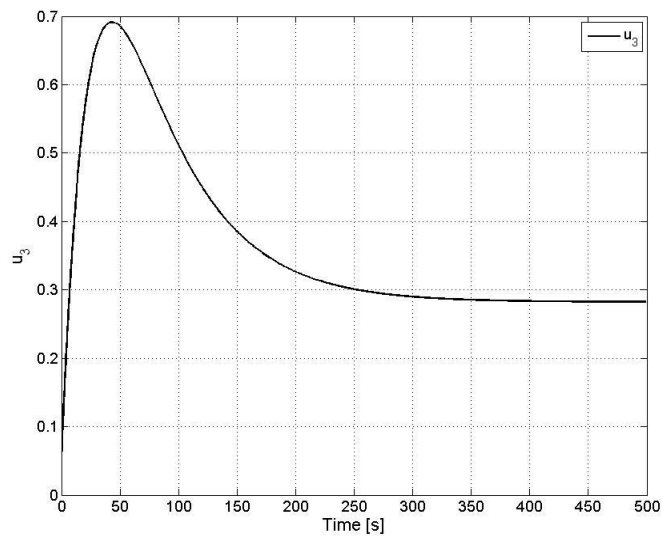


Fig. 7 MPC signal u_1

**Fig. 8** MPC signal u_2 **Fig. 9** MPC signal u_3

6. CONCLUSION

This paper describes the application of a digital model predictive control (MPC) based on the Laguerre functions applied to the nonlinear system with multiple inputs and outputs. As a control object, the system with three reservoirs of different shapes and dimensions was used. The system is valve-controlled. Digital MPC has been designed and implemented, and satisfactory results have been obtained regarding the robustness of the proposed controller.

For the future work, it is planned to use the other orthogonal, almost orthogonal and quasi-orthogonal functions of the Legendre type. In order to cope with the disturbance, a combination of MPC and sliding mode control should be considered. This combination could use the good features of each of the aforementioned control algorithms to obtain an optimal control law.

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