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Testing of a Coal-Fired Diesel Power Plant

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Contractor:

Arthur D. Little, Inc. 20 Acorn Park Cambridge, MA 02140

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Conference Dates: October 27-29, 1992

Conference Sponsor:

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Testing a Coal-Fired Diesel Power Plant

CONTRACT INFORMATION

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Contract Number	DE-AC21-88MC-25124
Contractor	Arthur D. Little, Inc. 20 Acorn Park Cambridge, MA 02140 (617) 498-5000
Contract Project Manager	Robert P. Wilson
Principal Investigators	Eric N. Balles (Arthur D. Little) Kam Rao (Cooper-Bessemer) Fred Schaub (Cooper-Bessemer) Jack Kimberley (AMBAC) Karen R. Benedek (Arthur D. Little) Charles E. Benson (Arthur D. Little) Dan Itse (PSI Technology)
METC Project Manager	W. Cary Smith
Period of Performance	March 1, 1988 to September 30, 1993

Schedule and Milestones

Program Schedule

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Phase II Develop Test Plan Design/Fabricate Develop Components							•		E	Ē									-		•		•
Phase III Exploratory Tests Modify Facility Performance Test Evaluation Handbook					· · · · · · · · · · · · · · · · · · ·						-						LSB-1 E	ngine "	Fest	Te	d Plan		• Test 0 hr)] []

OBJECTIVES

Development of coal-burning heat engines for 2 - 50 MW modular stationary power applications requires significant technical advances in engine design, fuel preparation and handling, durable components, and emission controls. The overall goal of this DOE/METC funded program is to advance the state-of-the-art for stationary coalfired diesel engines to the next plateau of technical readiness and thus provide the springboard to commercialization. The objective in this phase of the program is to develop practical, durable engine system components which are compatible with engine grade coal slurry fuel and are also capable of achieving low emission levels. These components have been integrated into a 1.8 MW coal-fired power plant which has been successfully run on coal fuel and is being prepared for a 100 hour (continuous) proof-of-concept (POC) demonstration test in 1993.

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BACKGROUND INFORMATION

The POC coal-fired power plant consists of a Cooper-Bessemer LSC-6 engine (15.5 inch bore, 22 inch stroke) rated at 400 rev/min and 208 psi bmep producing approximately 1.8 MW of power. The power plant is fueled with 'engine grade' coal slurry which has been physically cleaned to an ash level of approximately 1.5 to 2% (dry basis) and has a mean particle size of approximately 12 micron. CWS is injected directly into the combustion chamber through a fuel injector (one per cylinder) which was designed and developed to be compatible with the fuel. Each injector is fitted with a 19 orifice nozzle tip made with sapphire inserts in each orifice. The combustion chambers are fitted with twin diesel pilot injectors which provide a positive ignition source and substantially shorten the ignition delay period of the CWS fuel. Durable coatings (typically tungsten carbide) are used for the piston rings and cylinder liners to reduce wear rates. The emission control system consists of SCR for NO, control, sodium sorbent injection for SO, control, and a cyclone plus baghouse for particulate capture. The cyclone is installed upstream of the engine turbocharger which helps protect the turbine blades.

PROJECT DESCRIPTION

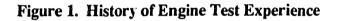
Arthur D. Little, Inc., is leading a five-year. three-phase component development program with Cooper-Bessemer as the principal subcontractor responsible for development testing. Cooper-Bessemer is the premier manufacturer of 300-450 rpm engines in the United States, and is the only U.S. manufacturer of 300-450 rpm engines which has successfully operated a test engine on coal water slurry. Phase I, the R&D effort, which is complete, explored candidate alternative solutions to the most pressing design issues. Laboratory, bench-top and engine experiments were conducted as appropriate. The Phase I research efforts were selected with the benefit of results and discoveries from the 1986-1987 METC-sponsored Cooper-Bessemer/ADL coal diesel project. During Phase II, we developed working hardware prototypes for each component technology (clean coal fuels, injection, combustion, wear and emissions). Cooper, PSI, CQ, Inc., EI, SwRI, AMBAC, and Battelle carried out experimental activities in each component technology area.

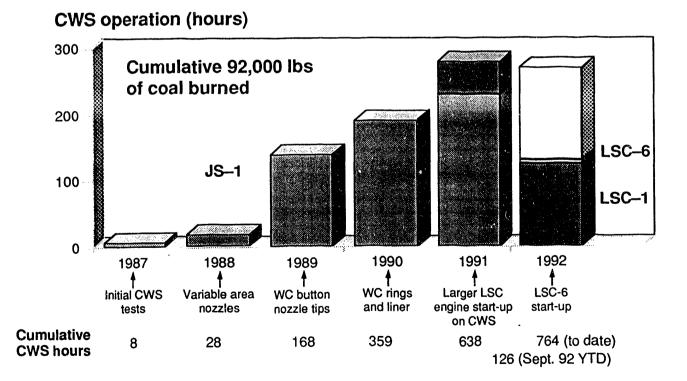
RESULTS

Over the past several years, significant progress has been made in developing coal engine components in preparation for the 100 hour POC test. A total of more than 685 hours of CWS operation have been logged on Cooper-Bessemer engines. Over 560 hours of CWS testing have been accumulated on a single cylinder research engine (JS-1) to develop fuel specifications, durable components and combustion system configurations. An additional 125 hours of CWS testing have been accumulated on the full-scale LSC-1 engine (one cylinder on coal). The LSC-6 (all six cylinders on coal) has also been successfully run on CWS. These engine tests have provided valuable insights into coal diesel combustion phenomena and engine component design. Table 1 summarizes the key development efforts for the three test engines. Figure 1 illustrates the history of Cooper-Bessemer test experience from 1987 to date.

JS1	13 inch bore 16 inch stroke single cylinder	 Exploratory CWS Tests Fuel handling Injector system development Durable components Coal diesel combustion phenomenon
LSC-1	15.5 inch bore 22 inch stroke 1 cylinder CWS 5 cylinders DF2 436 bhp/cylinder	 Full-Scale Development Tests Injection system Emission controls Combustion optimization
LSC-6	15.5 inch bore 22 inch stroke 6 cylinders CWS 2,616 bhp	 Proof-of-Concept Demo 6-cylinder fuel system Additional combustion optimization 100-hour POC test

Table 1. Coal-Fired Power Plant Development





LSC Configuration

In the past year, Cooper-Bessemer has adapted their six cylinder LS series engine to operate one cylinder on coal with the other five on diesel fuel. The research results and component designs from the single cylinder JS-1 tests were scaled-up and translated to the full-scale LS engine. Approximately 125 hours of CWS testing on the LSC-1 were completed. The research results were used to develop and refine the full-scale fuel injection and combustion system before switching over to all six cylinders on coal.

Cooper then undertook the substantial task of converting the LS to six cylinder coal operation. A number of major engine components were redesigned to accommodate the coal engine requirements. Substantially larger jerk pumps were installed because larger fuel flow rates are required with CWS compared to DF2. A new cam shaft was installed to obtain the desired fuel injection rate and to carry the increased load from the larger jerk pumps. The cylinder block was also redesigned to accommodate the jerk pumps and larger cam shaft size. Based on the LSC-1 findings, the initial LSC-6 configuration consisted of:

- injection cam: CB LSC-16-1C 'fast rate' cam
- injection jerk pump: L'Orange 36mm plunger and barrel
- nozzle tip: 19 x 0.633mm diameter holes with sapphire inserts
- nozzle spray angle: 140 degree
- injection timing: 23 degrees BTC port closure
- ignition aid: twin diesel pilot injectors at 120mm³/stroke/injector (approximately 4 to 5% of the total fuel energy).

Coal Water Slurry Fuel

Engine grade CWS fuel for these tests was supplied by CQ Inc. The coal was physically cleaned to approximately 1.8% ash with a 12 micron mean particle size. The slurry had a solids content of 48 percent which resulted in a heating value of 7300 Btu/lb. CWS properties are summarized in Table 2.

Ash	1.8% (wt% dry basis)
Sulfur	0.6% (wt% dry basis)
Volatile	30% (wt% dry basis)
Fixed carbon	61% (wt% dry basis)
Mean particle size	12 microns
Solids loading	48% (nominal)
CWS heating value	7,300 Btu/lb

Table 2. Coal Water Slurry Fuel

LSC Engine Performance

The LSC-6 underwent an extended break-in procedure and was then successfully tested on coal water slurry fuel. The break-in procedure was designed specifically for the hard in-cylinder components (piston rings and cylinder liners). Detailed borescope inspection of the tungsten carbide coated liners during and after the break-in run revealed nothing unusual. Transition from diesel operation (the engine is started and warmed-up on diesel) to CWS operation was demonstrated to be extremely smooth and quick. All six cylinders can be switched at once with the engine at full speed (400 rev/min) and 75% load (150 psi bmep). Transition from full diesel to full CWS takes place in less than 15 seconds once coal reaches the injectors. Other than a slight audible change in engine sound, the only outward indications the engine is running on CWS are the substantial reductions in NO_x and CO emission levels (3 fold and 6 fold reduction respectively).

Table 3 summarizes the initial LSC-6 test results. The data was taken at full speed (400 rev/min) and 3/4 load (150 psi BMEP) yielding approximately 1900 brake horsepower. Peak firing pressure and exhaust temperature were well within design limits. The measured engine efficiency (7500 Btu/bhp•hr) was considered acceptable for the initial tests. It is important to note that initial combustion optimization was conducted on the LSC-1 and that additional optimization will be undertaken on the LSC-6. Table 3 also lists DF2 performance data taken under the same operating conditions using the CWS injection system. Diesel performance under conditions optimized for DF2 (nozzle hole size, injection timing, etc.) would be different.

	CWS (initial results)	DF2 (baseline)*
Speed (rev/min)	400	400
BMEP (psi)	150	150
Power (bhp)	1,890	1,890
Peak Firing Pressure (psi)	900	1,320
Cylinder Exhaust Temp (°F)	870	930
Specific Fuel Consumption (Btu/bhp•hr)	7,500 (LHV)	7,300 (LHV)
NO _x (ppm)	460	1,180
CO (ppm)	300	2,100
CO ₂ (%)	7.5	5.8
O ₂ (%)	11.5	12.3

Table 3.	Recent L	SC-6	Engine	Performance	Results
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* Using CWS injection system (i.e. not optimized for diesel fuel)

Emission Control System Performance

The emission control system (ECS) was successfully tested at full flow with the engine operating on diesel fuel. Work thus far has focused on performance testing of the SCR and sorbent injection systems, operational testing of the mechanical components (ID fan, dampers, and rotary air locks), performance testing of the heat exchanger, and evaluation of the system controls and safety interlocks.

The initial ECS performance tests have shown

that the target reductions for NOx and SO_2 have been readily achieved. The SCR system alone is capable of 80 to 90% reduction in NOx. The sorbent injection process also reduces NOx emission. The level of reduction has been measured at 20 to 40% which corresponds to about 4 to 8% of the engine exhaust NOx emissions. Measured SO2 reduction at a Na₂/SO₂ mole ratio of 0.8 and a baghouse inlet temperature of 360 F was in the 75 to 80% range. These test results, summarized in Table 4, show that the integrated system is capable of low emission levels which also meet NSPS standards for electric utilities.

	Engine Emissions	Target Reduction	Test Results	Integrated System Emissions Based on Test Results	NSPS for Electric Uti- lities
NO _x	1.3 lb/MM Btu	80% (SCR)	90% (SCR plus sorbent)	0.13 lb/MM Btu	0.6 lb/MM Btu
SO	1.5 lb/MM Btu	70%	80% (does not include credit for coal clean-ing)	0.30 lb/MM Btu	1.2 lb/MM Btu (and 90% reduction in potential SO ₂)
Particulates 1.75% ash 1% unburned C	2.5 lb/MM Btu	97.5%	(99%) Bag filter not tested yet	(0.025 lb/MM Btu)	0.03 lb/MM Btu

Table 4. Latest Emission Control System Performance Results

FUTURE WORK

The LSC-6 is being prepared for the 100 hour (continuous) proof-of-concept demonstration test which is scheduled for mid-1993. Additional combustion optimization and emission control tests will be conducted to fine-tune the integrated system. Several 25 and 50 hour tests are scheduled prior to the 100 hour test to ensure refueling procedures are debugged. The program will culminate in a 100 hour continuous demonstration of the coal-fired diesel power plant.

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