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**Development of Standardized Air-Blown Coal Gasifier/  
Gas Turbine Concepts for Future Electric Power Systems  
Volume II  
Appendix A: Fixed Bed Gasifier and Sulfur Sorbent  
Regeneration Subsystem Computer Model Development**

**Final Report**

**E. Blough  
W. Russell  
J.W. Leach**


August 1990

Work Performed Under Contract No.: DE-AC21-89MC26291

For  
U.S. Department of Energy  
Office of Fossil Energy  
Morgantown Energy Technology Center  
Morgantown, West Virginia

By  
CRS Serrine, Inc.  
Power Division  
Greenville, South Carolina

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**August 1990**

MODELING OF STANDARDIZED  
INTEGRATED COAL GASIFICATION  
COMBINED CYCLE POWER PLANTS

Technical Report

Submitted to

CRS Serrine Incorporated

August 15, 1990

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## ABSTRACT

Computer models have been developed for evaluating conceptual designs of integrated coal gasification combined cycle power plants. An overall system model was developed for performing thermodynamic cycle analyses, and detailed models were developed for predicting performance characteristics of fixed bed coal gasifiers and hot gas clean up subsystem components. The overall system model performs mass and energy balances and does chemical equilibrium analyses to determine the effects of changes in operating conditions, or to evaluate proposed design changes. An existing plug flow model for fixed bed gasifiers known as the Wen II model was revised and updated. Also, a spread sheet model of a zinc ferrite sulfur sorbent regeneration subsystem was developed. Parametric analyses were performed to determine how performance depends on variables in the system design. The work was done to support CRS Sirrine Incorporated in their study of standardized air blown coal gasifier gas turbine concepts.

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# 1 INTRODUCTION

There is a need for cost effective and environmentally acceptable power generation systems that can utilize our nations vast coal resources. Thus, integrated coal gasification combined cycle (IGCC) power plants are now under intense study because of their potential for accomplishing clean and cheap conversion of energy in coal. IGCC systems that burn low BTU coal gas produced in air blown fixed bed gasifiers appear to have economic advantages. The hot pressurized coal gas from the gasifier is cleaned prior to combustion in a gas turbine. The volume of the coal gas that is cleaned is much less than the volume of the stack gases that must be cleaned in a conventional plant. Also, the sulfur in the coal gas is in the form of hydrogen sulfide which can be removed more efficiently than can sulfur dioxide in flue gases. The overall efficiency of the combined cycle plant is expected to be in excess of 40%. Thus, carbon dioxide emissions as well as sulfur dioxide omissions will be reduced.

Research and development on IGCC systems has been in progress for the last two decades, and several demonstration plants have been built. The demonstration plants in the United States have operated on medium BTU gas produced by oxygen blown entrained flow gasifiers. Plants are now being designed to burn cheaper low BTU gas from air blown fixed bed gasifiers, and a demonstration plant that will process about 1000 tons of coal per day will be built near Tallahassee Florida in the early 1990's.

The work documented in this report was done in support of a larger study conducted by CRS Serrine Incorporated for the United States Department of Energy on low cost standardized IGCC systems designs. Our work is concerned with the development of computer models. We have developed an overall IGCC system model for performing thermodynamic cycle analysis, and have also worked towards the development of detailed computer models of two of the major components in the IGCC system.

The overall system model is useful for evaluating the effects of changes in operating conditions or of changes in subsystem designs on the net plant efficiency and power output. We have revised and updated an existing Fortran model of a fixed bed coal gasifier to gain a

better understanding of the gasification process and to determine the effects of variations in coal properties such as moisture content and particle size on the plant performance. We have also developed a new spread sheet model for analyzing the regeneration and sulfur recovery process in a hot gas cleanup system that employs zinc ferrite to remove hydrogen sulfide from the coal gas.

Two computer models have been developed for analyzing the coal gasifier. One model simulates the detailed processes that occur within the gasifier. It contains chemical reaction rate equations, conservation equations, heat transfer correlations, a devolatilization model, and other information needed for analyzing the gasifier from basic principles. It is useful for predicting local temperatures within the gasifier, carbon conversion efficiency, product gas composition, and other detailed information about the gasification process. This model originated at West Virginia University. We have made extensive modifications in the present work. A second computer model of the coal gasifier was developed for evaluating IGCC system designs. This model performs mass and energy balances, and is capable of predicting the product gas composition based on chemical equilibrium assumptions. It can also accept experimentally determined gas compositions as input. It is useful for predicting how the gasifier operating conditions affect the overall system performance. For example, it could be used to determine how the overall power plant efficiency changes when the steam flow to the gasifier is reduced.

Hot gas cleanup will contribute to the success of the IGCC. Conventional processes for removing impurities from coal gas must operate at low temperatures, and the plant cycle efficiency would suffer if the gas had to be cooled for cleaning before it is burned in the combustor. The zinc ferrite sorbent hot gas cleanup system is a promising alternative to conventional scrubber technology. However, the zinc ferrite is expensive and must be regenerated when it becomes charged with sulfur. The sorbent regeneration process and sulfur recovery subsystem should be integrated into the overall plant design. We have developed a computer model to gain an understanding of how this should be done, and have done parametric analysis to determine how the required gas flows depend on variables in the subsystem design.

## 2. IGCC POWER PLANT SIMULATION

This section provides a detailed description of a computer model that was developed for evaluating the overall performance of integrated coal gasification combined cycle (IGCC) plants. The model is useful for evaluating the effects of changes in the system configuration (or of improvements in individual component performance) on the net cycle efficiency and power output. It performs thermodynamic analysis of the individual components of the system and accounts for mass and energy balances. Also, in the gasifier and combustor where chemical reactions occur, the model can perform chemical equilibrium calculations to determine the composition of the product gas streams. It does not account for reaction kinetics, but gives the user the option of inputting the product gas composition when chemical equilibrium calculations are not acceptable. Whether the product gas compositions are computed by the model or provided by the user, the model computes product gas flows and temperatures to insure that mass and energy are conserved.

The IGCC plant employs a fired heat recovery steam generator (HRSG) which provides low pressure steam for the coal gasification and high pressure steam for the Rankine cycle power plant. The computer model contains steam tables and source code for analyzing the HRSG and Rankine cycle plant.

### 2.1 IGCC Plant Description

An Integrated Gasification Combined Cycle Power Plant using a GE MS7001 EA gas turbine utility power plant computer model is illustrated in Figure 2.1. Air flows into the gas turbine compressor at a mass flow rate of about 600 lbm/sec. The air is pressurized in the compressor to a pressure of about 200 psi, and the temperature increases to around 800° F. This compressed air is divided into three streams. About 5 percent of the air by-passes the combustor and is used for cooling purposes in the turbine. About 9 percent of the air is sent to a booster compressor for use in the gasification process. The remaining air is sent to the combustor.



The air that is sent to the gasifier is increased in pressure to 300 psi, and increased in temperature to 1070 °F. In the gasifier, coal, steam, water, and air are mixed and react to produce a low BTU gas that will be burned in the gas turbine combustor. This raw gas leaving the gasifier is purified in a hot gas cleanup unit before it is throttled to the pressure of the combustor. The hot gas cleanup unit is described in detail in Section 4. Our model contains an optional turbine that could derive power from the high pressure coal gas that flows from the gasifier. This turbine will not be employed in the actual IGCC system. It was included in the model so that we could make computations to determine how the overall system efficiency is affected by the throttling of the high pressure coal gas.

The low BTU gaseous fuel is mixed with air and burned in the gas turbine combustor. The mass flow rate exiting the combustor is about 600 lbm/sec and 2100 °F. The pressure at this location is close to 175 psi. These gases now expand through the turbine area creating power.

Our model contains equations for a fired heat recovery steam generator. Originally, this component was added to the model so that we could evaluate the effect of burning coal fines. It appears that this is undesirable because of the sulfur in the fines. However, it may prove to be advantageous to burn clean coal gas in the heat recovery steam generator. We have arbitrarily made example computations for the case when the exhaust gas is heated to 1250 F before it is used to raise steam. In this case the mass flow rate of the exhaust gas is almost 650 lbm/sec. This flow is sent to the heat recovery part of the Integrated Gasification Combined Cycle.

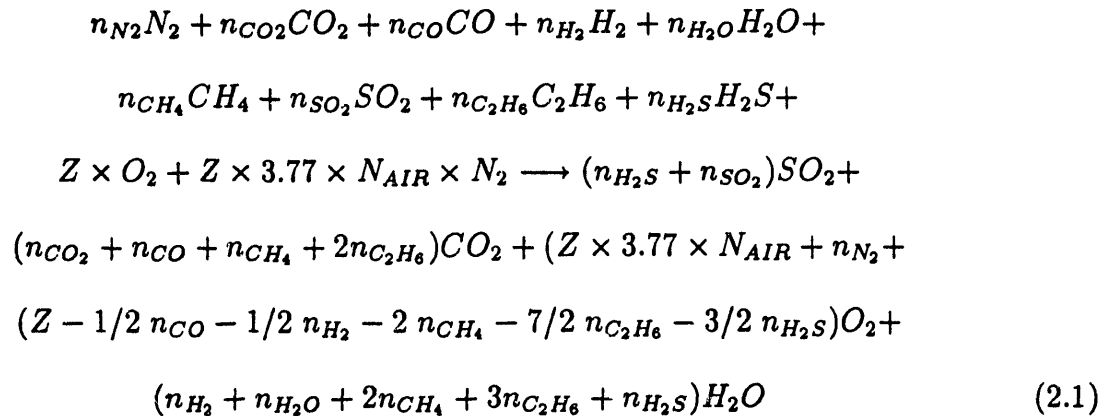
## 2.2 Combustor Model Using Equilibrium Constants

The computer model of the combustor determines the firing temperature and the combustor product gas composition for given fuel gas properties and fuel flow rates. It can make calculations based on complete combustion while accounting for combustor losses, or it can do chemical equilibrium calculations to determine the equilibrium fractions of carbon monoxide, hydroxyl, nitric oxide, and other radicals in the combustor product gases. One



way of determining the combustion temperature is to use an energy balance of the products and reactants coupled with equilibrium equations that allow the extent of reaction to be determined as a function of temperature.

For example, Figure 2.1 shows that the reactants for the combustor are the products of the gasifier from stage 12 and the air from compressor 1 at stage 14. Consequently, a straight forward calculation can be performed. Below is the complete combustion equation for the reaction of  $N_2$ ,  $O_2$ ,  $CO_2$ ,  $CO$ ,  $H_2O$ ,  $H_2$ ,  $CH_4$ ,  $C_2H_6$ ,  $H_2S$ , and  $SO_2$ .

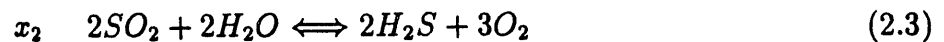


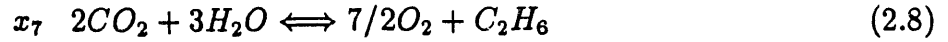
In the above equation " $N_{AIR}$ " may be chosen to be either one or zero. If the user chooses one, the computer model simulates the combustion of the products with air. If the user chooses zero for this parameter the computer model simulates the combustion with pure oxygen.

The extent of the reaction that will occur for each of the products can be determined from chemical equilibrium analyses. Below are the equilibrium equations for the reactions in the combustor.

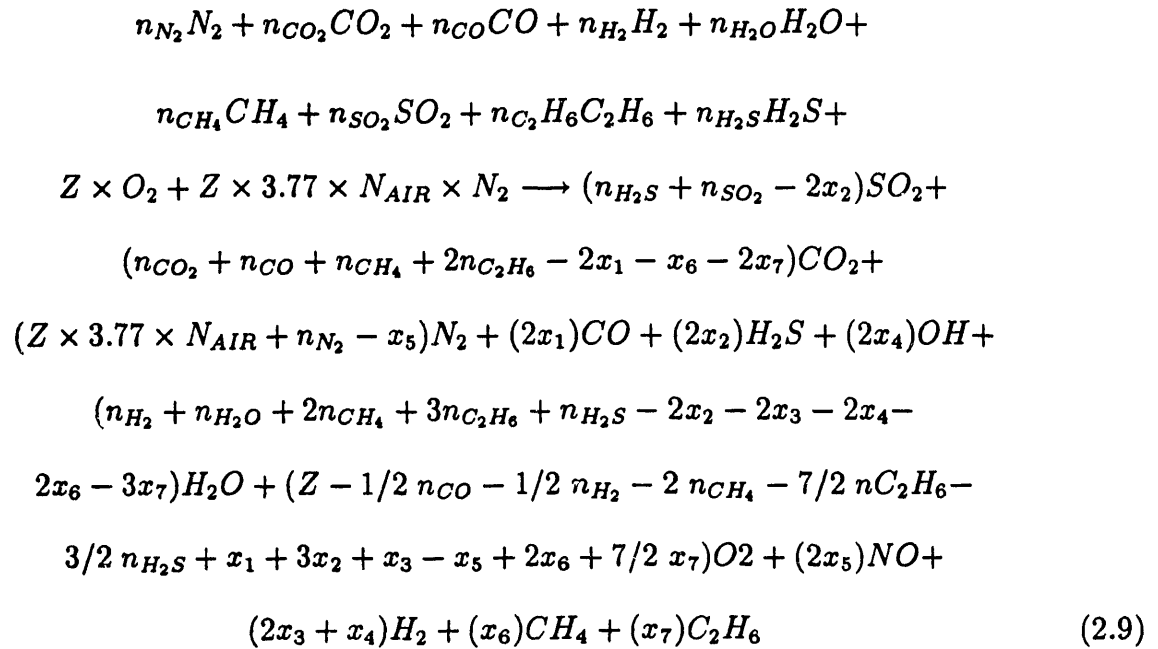
**Balancing Constant**

**Equation**





where ' $x_N$ ' represents the extent of reaction that each of the above equations will experience at a given temperature. The reaction equation for incomplete combustion is:



The extent of each reaction may be calculated at any temperature and pressure by solving equilibrium equations. In general, for the equation



The equilibrium equation can be written as:

$$\ln \frac{(Y_C P/P_o)^{v_D} (Y_D P/P_o)^{v_D}}{(Y_A P/P_o)^{v_A} (Y_B P/P_o)^{v_B}} = \frac{v_C G_C + v_D G_D - v_A G_A - v_B G_B}{RT} \quad (2.11)$$

By writing the equilibrium equations for our combustion problem, we obtain the following:

For equation  $2CO_2 \rightleftharpoons 2CO + O_2$ :

$$\ln \frac{(Y_{CO}P/100)^2(Y_{O_2}P/100)^1}{(Y_{CO_2}P/100)^2} = \frac{2G_{CO_2} - 2G_{CO} - G_{O_2}}{RT} \quad (2.12)$$

For equation  $2SO_2 + 2H_2O \rightleftharpoons 2H_2S + 3O_2$  :

$$\ln \frac{(Y_{H_2S}P/100)^2(Y_{O_2}P/100)^3}{(Y_{SO_2}P/100)^2(Y_{H_2O}P/100)^2} = \frac{2G_{SO_2} + 2G_{H_2O} - 2G_{H_2S} - 3G_{O_2}}{RT} \quad (2.13)$$

For equation  $2H_2O \rightleftharpoons 2H_2 + O_2$ :

$$\ln \frac{(Y_{H_2}P/100)^2(Y_{O_2}P/100)^1}{(Y_{H_2O}P/100)^2} = \frac{2G_{H_2O} - 2G_{H_2} - G_{O_2}}{RT} \quad (2.14)$$

For equation  $2H_2O \rightleftharpoons H_2 + 2OH$ :

$$\ln \frac{(Y_{H_2}P/100)^1(Y_{OH}P/100)^2}{(Y_{H_2O}P/100)^2} = \frac{2G_{H_2O} - 2G_{H_2} - 2G_{OH}}{RT} \quad (2.15)$$

For equation  $N_2 + O_2 \rightleftharpoons 2NO$ :

$$\ln \frac{(Y_{NO}P/100)^2}{(Y_{N_2}P/100)^1(Y_{O_2}P/100)^1} = \frac{G_{N_2} + G_{O_2} - 2G_{NO}}{RT} \quad (2.16)$$

For equation  $CO_2 + 2H_2O \rightleftharpoons 2O_2 + CH_4$ :

$$\ln \frac{(Y_{O_2}P/100)^2(Y_{CH_4}P/100)^1}{(Y_{CO_2}P/100)^1(Y_{H_2O}P/100)^2} = \frac{G_{CO_2} + 2G_{H_2O} - 2G_{O_2} - G_{CH_4}}{RT} \quad (2.17)$$

For equation  $2CO_2 + 3H_2O \rightleftharpoons 7/2O_2 + C_2H_6$ :

$$\ln \frac{(Y_{O_2}P/100)^{7/2}(Y_{C_2H_6}P/100)^1}{(Y_{CO_2}P/100)^2(Y_{H_2O}P/100)^3} = \frac{2G_{CO_2} + 3G_{H_2O} - 7/2G_{O_2} - G_{C_2H_6}}{RT} \quad (2.17)$$

An energy balance was used to determine the combustion temperature. This balance included the enthalpies of the reactants, enthalpies of the products, and a combustion efficiency parameter.

Because the combustion temperature is a function of the extent of reaction which in turn is a function of temperature, a Newton-Raphson iterative process is used. With the Newton-Raphson method accurate solutions can be obtained with as few as five iterations.

Although the above analysis is included in our model, and is useful for determining amounts of disassociation that may occur within a combustion process, it is not actually needed for determining the temperature of the reaction, the main objective of this part of the program. There is excess air in the gas turbine combustor, and the firing temperatures are low enough that little dissociation occurs. Thus it is acceptable to assume that complete combustion occurs. For this reason later versions of this program only calculated the complete combustion temperature, which was found to be extremely close to temperatures calculated in the analyses using equilibrium equations.

It should be noted that the overall system model described here assumes that the combustor design is adequate for accomplishing complete combustion of low BTU coal gas. Separate computer models which account for turbulent mixing of the fuel and air, chemical kinetics, radiation heat transfer, and other details of reacting flows are needed to predict whether a combustor design is indeed adequate. Two computer models of gas turbine combustors were evaluated in a separate task of the IGCC system study.

### **2.3 Gasifier Model Using Equilibrium Constants**

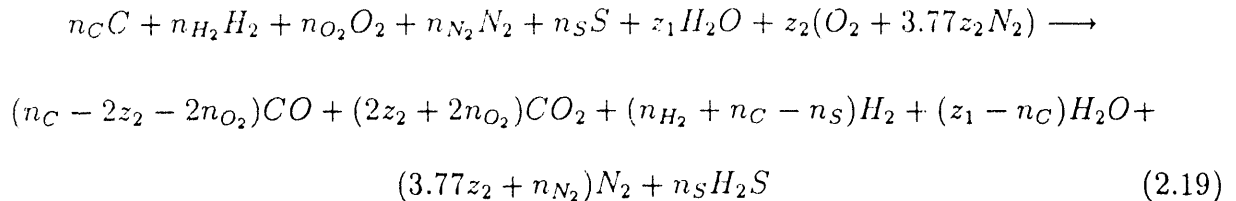
A detailed model of a fixed bed coal gasifier was developed separately, and is described in Section 3. It is useful for predicting allowable coal feed rates for given gasifier dimensions, local temperatures within the gasifier (which is needed to predict whether ash fusion is a potential problem), carbon conversion efficiency, and other specific information about the coal gasifier. A less detailed model is needed for the overall system analysis. The

requirements in this case are that the model must conserve mass and energy, and must account for the chemical composition of the coal gas. Chemical equilibrium analysis are not as useful for predicting gas compositions in the gasifier as they are in the combustor because temperatures are lower in the gasifier. Chemical equilibrium may not be approached because the reaction rates are slow.

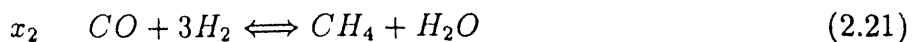
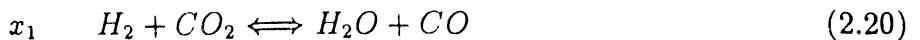
It was also found that equilibrium analysis tended to under predict the amount of heavier hydrocarbons in the coal gas. The result of this is that the model tended to underpredict the heating value of the coal gas and overpredicted the gas exit temperatures. The model predicted that too little of the energy originally in coal remained as stored chemical energy in the product gas and that too much was stored as sensible energy in the gas.

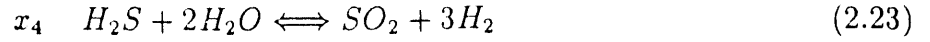
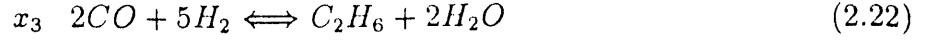
Nevertheless, the model predictions base on equilibrium analysis are adequate for the purpose of the study. For example, based on chemical equilibrium analyses, the model predicted that the lower heating value of the dry product gases is 163 BTU/scf. Based on experimental gas compositions, the heating value is 168 BTU/scf. To compensate for this error, the model predicted that the gas temperatures for the chemical equilibrium case is about 300 °F higher than for the case with experimental gas composition. The user has the option of inputting the experimental gas composition when it is known.

The first step in developing the equation for the chemical equilibrium analysis is to create a complete gasification equation from the known reactants. The equation is:

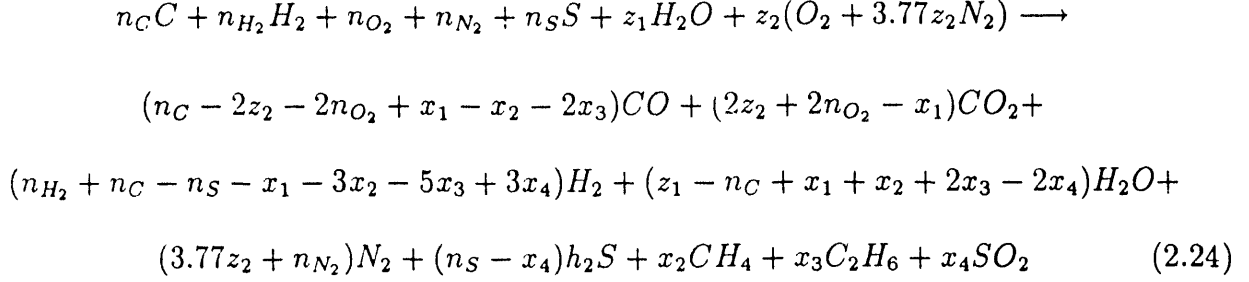


The next step is to write the important equilibrium equations needed to determine the amounts of  $O_2$ ,  $N_2$ ,  $CO$ ,  $CO_2$ ,  $H_2$ ,  $H_2$ ,  $CH_2$ ,  $C_2H_6$ ,  $H_2O$ ,  $SO_2$ , and  $H_2S$  as a function of temperature and pressure. The reactions that are considered are:





where  $x_1$ ,  $x_2$ ,  $x_3$ , and  $x_4$  signify the extent of the reaction of each of the above equations. These equations can now be combined with equation 19 to give:



Equation 24 is simply a balanced reaction equation in which all of the components that are of significance in the coal gas appear as products. The values of  $x_1$  through  $x_4$ , which specify the amount of each component, are obtained by minimizing the Gibbs energy of the product gas mixture. The following equations establish the conditions for which the Gibbs energy is a minimum:

$$\frac{dG}{dx_1} = \frac{DG}{dx_2} = \frac{dG}{dx_3} = \frac{dG}{dx_4} = 0 \quad (2.25)$$

By writing an equation for the Gibbs energy of the mixture of produced gases in Equation 2.24 and differentiating with respect to  $x_1$  through  $x_4$  we obtain the following:

$$\ln \frac{(Y_{H_2O} P/P^0)^1 (Y_{CO} P/P^0)^1}{(Y_{H_2} P/P^0)^1 (Y_{CO_2} P/P^0)^1} = \frac{G_{H_2} + G_{CO_2} - G_{H_2O} - G_{CO}}{RT} \quad (2.26)$$

$$\ln \frac{(Y_{CH_4} P/P^0)^1 (Y_{H_2O} P/P^0)^1}{(Y_{CO} P/P^0)^1 (Y_{H_2} P/P^0)^3} = \frac{G_{CO} + 3G_{H_2} - G_{CH_4} - G_{H_2O}}{RT} \quad (2.27)$$

$$\ln \frac{(Y_{C_2H_6} P/P^0)^1 (Y_{H_2O} P/P^0)^2}{(Y_{CO} P/P^0)^2 (Y_{H_2} P/P^0)^5} = \frac{2G_{CO} + 5G_{H_2} - G_{C_2H_6} - 2G_{H_2O}}{RT} \quad (2.28)$$

$$\ln \frac{(Y_{SO_2} P/P^0)^1 (Y_{H_2} P/P^0)^3}{(Y_{H_2S} P/P^0)^1 (Y_{H_2O} P/P^0)^2} = \frac{G_{H_2S} + 2G_{H_2O} - G_{SO_2} - 3G_{H_2}}{RT} \quad (2.29)$$

The mole fractions of the product gases may be expressed in terms of  $x_1$  through  $x_4$  using Equation 2.24. A Newton Raphson search procedure is then employed to solve Equation 2.26 through Equation 2.29 for  $x_1$  through  $x_4$ . However a fifth variable (temperature)

must be considered because the Gibbs energies in Equation 2.26 through Equation 2.29 are temperature dependent.

The product gas temperature may be determined from the energy equation which is written below in terms of the enthalpies of the gaseous components, heat losses, change of energies of the ash, and the heat of formation of the coal.

$$n_{air}h_{air} + n_{liq\ water}h_{liq\ water} + n_{stream}h_{stream} + m_{coal}h_{f\ coal} =$$

$$g_{loss} + \sum n_{prod}h_{prof} + m_{ash}(E_{ash\ @2} - E_{ash\ @1}) \quad (2.30)$$

Once again, a Newton-Raphson method was used to solve the five non-linear equations. However, the solution to the equations for the gasifier is not as straight forward as the solution to the equation for the gas turbine combustor. The gasification process is at a lower temperature with much smaller amounts of oxygen. Convergence was much more difficult. In particular, determining the values of  $x_1$  through  $x_4$  was very tedious. Because the gasification reactions are done in an oxygen poor environment, the amount of oxygen in the product gas is extremely low. Consequently, while iterating towards the solution, the amount of oxygen might be temporarily calculated to be less than zero. However, before another iteration could compensate for this error, the program would crash. For the above reason, initial guesses in the Newton Raphson search could not be totally random.

## 2.4 Compressor and Turbine Model

Standard equations for the compressor and the turbine are used in the computer model. The work needed for the compressors or the work produced by the turbines are calculated using the following equations:

$$Compressor\ Work = \dot{m}_{air} * C_{p_{air}} * \Delta Temp \quad (2.31)$$

$$Turbine\ Work = \dot{m}_{products} * C_{p_{products}} * \Delta Temp \quad (2.32)$$

The following are the compressor and turbine equations used to determine exiting temperatures given a prescribed pressure ratio.

For the Turbine:

$$T_{out} = T_{in} * \left( \frac{P_{out}}{P_{in}} \right)^{\left[ \frac{turbine\ eff * (k_{prod} - 1)}{k_{prod}} \right]} \quad (2.33)$$

For the Compressor:

$$T_{out} = T_{in} * \left( \frac{P_{out}}{P_{in}} \right)^{\left[ \frac{(k_{air} - 1)}{compressor\ eff * k_{air}} \right]} \quad (2.34)$$

## 2.5 Computer Model of Heat Recovery Cycle

The gas turbine exhaust is at a low pressure (approximately atmospheric) but at a high temperature (approximately 1000 °F). By using heat exchangers to cool the exhaust gas while heating water to make steam, waste energy in the low pressure exhaust gas can be rendered into useful energy in high pressure steam.

For the heat recovery steam generator (HRSG) and bottoming cycle, the pressures at every point in the cycle are set to optimize the cost and efficiency of the overall plant. Other knowns include the temperature of the exhaust gas entering the HRSG, the mass flowrate of the exhaust gas, the heat exchanger sizes, and the turbine and pump efficiencies. The temperature of the steam leaving the superheater is set by the temperature of the exhaust gas. From all the given information, the properties and mass flow rates at each point on the cycle can be determined.

In our bottoming cycle model, the starting and ending point for the calculations is at the condenser. First, the properties of water leaving the condenser are determined at saturated conditions. Equations for the pumps are then solved to determine the properties of the feedwater entering the boiler. An iterative process is used to determine both the properties of the gas and steam on either side of the superheater and the high pressure boiler/economizer. Finally, the properties of the steam returning to the condenser are determined by solving



equations for the turbines. The calculations for each separate component of the system are described in detail below.

### 2.5.1 Pumps

The knowns for the pump calculations are the pump efficiency, the pressures on both sides of the pump, and the properties of the water entering the pump. Assuming incompressible flow, the following formula is used to determine the amount of work done by the pump per kilogram of feedwater.

$$W_{pump} = \frac{v \cdot (P_{exit} - P_{enter})}{\eta_p} \quad (2.35)$$

Also, the enthalpy of the exiting water is determined using the following formulas.

$$h_{exit} = W_{pump} + h_{enter} \quad (2.36)$$

The above procedure is repeated for each of the three pumps.

### 2.5.2 Boilers/Economizers and Superheaters

The knowns for the models of the boilers/economizers are the inlet exhaust gas temperature, the size of the boilers, and the inlet and exit properties of the steam.

In our model, the overall  $UA$  for the boiler is divided into two regions, the preheat region and the boiling region. Since the steam leaving the boilers is saturated at the design pressure, the properties for steam are known for both the entrance and the exit of the boiler. Also since the exhaust gas exit temperature is not known, the overall  $UA$  is written in terms of steam temperatures, and mass flow rates only. An energy balance is performed for each of these regions yielding the following energy equations.

$$q_1 = (\dot{m}c_p)_{gas} (T_{G1} - T_{G2}) = (UA)_1 \Delta T_{M1} \quad (2.37)$$

$$q_2 = (\dot{m}c_p)_{gas} (T_{G1} - T_{G1}) = (UA)_2 \Delta T_{M2} \quad (2.38)$$

where  $\Delta T_M$  is the logarithmic mean temperature for the region. Equations 2.37 and 2.38 are solved for their respective  $UA$  values, and are added together to give one equation for the overall  $UA$  value for the boiler/economizer.

$$UA = (\dot{m}c_p)_{gas} \left[ \frac{1}{1 - \frac{T_{S2} - T_{S1}}{T_{G1} - T_{G1}}} \ln \left( \frac{T_{G1} - T_{S2}}{T_{G1} - T_{S1}} \right) + \ln \left( \frac{T_{G1} - T_{S2}}{T_{G1} - T_{S2}} \right) \right] \quad (2.39)$$

where:

- $T_{G1} \equiv$  Exhaust gas inlet temperature (known)
- $T_{GI} \equiv$  Exhaust gas temperature at interface of regions (unknown)
- $T_{G2} \equiv$  Exhaust gas outlet temperature (unknown)
- $T_{S1} \equiv$  Water inlet temperature (known)
- $T_{S2} \equiv$  Steam outlet temperature (known)

The overall steady state energy equation for both regions is solved for the exhaust gas exit temperature,  $T_{G2}$ .

$$T_{G2} = T_{G1} - \frac{(\dot{m}c_p)_{water}}{(\dot{m}c_p)_{gas}} \left( T_{S2} - T_{S1} + \frac{h_{fg}}{c_{p,water}} \right) \quad (2.40)$$

where  $h_{fg}$  is the heat of vaporization for water. Also, an energy balance for the preheat region when solved for the exhaust gas boiling interface temperature provide the following equation.

$$T_{GI} = T_{G2} + \frac{T_{S2} - T_{S1}}{R} \quad (2.41)$$

where:

$$R \equiv \frac{(\dot{m}c_p)_{gas}}{(\dot{m}c_p)_{water}} \quad (2.42)$$

Equations 2.40 and 2.41 may be solved for the exhaust gas boiling interface temperature,  $T_{GI}$ .

$$T_{GI} = T_{G1} + \frac{1}{R} \frac{h_{fg}}{c_{p,water}} \quad (2.43)$$

Equations 2.39, 2.40, and 2.43 are combined to give a single equation for the overall  $UA$ .

$$\frac{UA}{(\dot{m}c_p)_{gas}} = \frac{1}{1 - R} \ln \left( \frac{T_{G1} - T_{S2} + \frac{1}{R} \frac{h_{fg}}{c_{p,water}}}{T_{G1} - T_{S1} - \frac{1}{R} \left( \frac{h_{fg}}{c_{p,water}} + T_{S2} - T_{S1} \right)} \right)$$

$$+ \ln \left( \frac{T_{G1} - T_{S2}}{T_{G1} - T_{S2} + \frac{1}{R} \frac{h_{fg}}{c_{p,water}}} \right) \quad (2.44)$$

In Eqn. 2.44 the only unknown is  $R$ , given by Eqn. 2.42.  $R$  is determined using the Newton-Raphson method, which means two trial values for  $R$  are required. To find these two trial values, the critical values for Eqn. 2.44 are determined. These critical values of  $R$  are defined as the values that make the denominators of either of logarithmic terms equate to zero. By equating these denominators to zero, the following expressions for  $R$  are determined.

$$R_{CRIT,1} = \frac{T_{S2} - T_{S1} + \frac{h_{fg}}{c_{p,water}}}{T_{G1} - T_{S1}} \quad (2.45)$$

$$R_{CRIT,2} = \frac{\frac{h_{fg}}{c_{p,water}}}{T_{G1} - T_{S2}} \quad (2.46)$$

The trial values of  $R$  must be larger than either of the critical values.  $R^*$  is defined as the larger of the two  $R_{CRIT}$  values.  $R^*$  is used in the following equations to determine the trial values for  $R$ .

$$R_1 = \frac{R^*}{1 - \exp \left( \frac{-UA}{(\dot{m}c_p)_{gas}} \right)} \quad (2.47)$$

$$R_2 = R_1 + \left[ \frac{UA}{(\dot{m}c_p)_{gas}} \right] [R_1 - R^*] \quad (2.48)$$

Once Eqn. 2.44 is solved for  $R$ , the flowrate of feedwater is determined from Eqn. 2.42. The feedwater flowrate is needed to compute the power produced by the Rankine cycle.

### 2.5.3 Turbines

The knowns for the turbine calculations are the turbine efficiency, the properties of steam entering the turbine, and the pressure drop across the turbine. The properties of steam are calculated at the outlet pressure and the inlet entropy. The work for the turbine is then calculated using the following formula.

$$W_t = \eta_t (h_{enter} - h_{exit,isentropic}) \quad (2.49)$$

The actual exit enthalpy is then calculated using the following formula.

$$h_{exit,actual} = h_{enter} - W_{TURBINE} \quad (2.50)$$

The properties of the exhaust steam are re-calculated using the exit pressure and the above enthalpy.

#### 2.5.4 Steam Tables

The steam table subroutine calculates the properties of steam from saturated liquid to superheated steam. The method used here is the one described in Reynolds [1] and Keenan [2]. A listing of the subroutines that make up the steam tables is included in the appendices.

### 2.6 Simulation Results

Five initial test cases were run to predict the overall system performance for the computer model. The Rankine bottoming cycle and heat recovery steam generator, which are described in Reference 1, were added to the model after the initial test cases were run. Separate cases were run to determine the performance of the combined cycle plant. The results of the initial test cases are summarized in Table 2.1 through Table 2.5.

As a first case the gas turbine was operated with methane fuel. A second run was made with a coal gas fuel when composition was determined experimentally. Table 2.1 shows that the cycle thermodynamic efficiency dropped from 32.14 % for methane fuel to 30.00 % for the coal gas fuel. Much of the loss in efficiency is due to the throttling of the high pressure coal gas as it flows from the gasifier to the combustor. This may be seen from the results of Table 2.3 in which a turbine was added to the system to replace the throttle valve. The efficiency improved to 31.45 % with the optional turbine. Two additional cases were run to establish the errors introduced by having the model compute the composition of the coal gas from equilibrium analysis. The results show that there is a negligible effect on the predicted overall system performance.

The overall system model is written such that it runs in an interactive mode. It prompts the user for needed information (such as the ultimate analysis of the coal), and provides a printout of the input data and the computed results. Table 2.6 shows the results of a typical run. It may be seen from the results in Table 2.6 that the fired HRSG and bottoming cycle increases the net shaft power by about 65 % and increases the thermodynamic efficiency (based on the lower heating value of the coal fed to the gasifier) to more than 41 %.

Table 2.1

\*\*\*\*\*  
\*\* COMMAND MENU \*\*  
\*\*\*\*\*

Please type the number or letter of the variable  
you wish to change and PRESS ENTER.

-----  
(TOGGLE VALUES)  
1) Fuel used in power system  
METHANE  
-----

(INPUT VALUES)  
3) Temperature of air at inlet (Location 1)..... 298.15 K  
4) Pressure of air at inlet (Location 1)..... 101.32 KPA  
5) Mass flow rate of air at inlet (Location 1)... 287.13 KG/SEC  
6) Fraction of air used to cool blades..... .050  
7) Efficiency of air compressors..... .860  
8) Efficiency of gas turbines..... .920  
9) Efficiency of combustor..... .950  
A) Efficiency of pipes..... .950  
B) Efficiency of generator and drive..... .960

1) STATISTICS  
(Amounts in moles - DRY & Sulfur FREE)  
CO2.... .00000  
CO.... .00000  
H2.... .00000  
CH4.... 1.00000  
CnHm... .00000  
N2.... .00000

location	ENGLISH UNITS		
	degrees F	psi	lbm/sec
1	76.67	14.69	633.00
2	76.58	14.68	633.00
3	719.52	171.80	633.00
4	719.52	163.21	633.00
11	80.00	21.75	11.01
12	300.20	155.05	11.01
13	719.52	163.21	601.35
14	719.52	155.05	601.35
15	2023.24	155.05	612.36
16	1614.37	70.84	612.36
17	1570.39	70.84	644.01
18	956.83	14.79	644.01

location	METRIC UNITS		
	degrees K	KPA	kg/sec
1	298.15	101.32	287.13
2	298.10	101.27	287.13
3	655.29	1184.81	287.13
4	655.29	1125.57	287.13
11	300.00	150.00	4.99
12	422.33	1069.29	4.99
13	655.29	1125.57	272.77
14	655.29	1069.29	272.77
15	1379.58	1069.29	277.77
16	1152.43	488.57	277.77
17	1128.00	488.57	292.12
18	787.13	102.00	292.12

WORK INPUT/OUTPUT (KW)  
 Work of FIRST COMPRESSOR -----> 106878.3  
 Work of SECOND COMPRESSOR -----> 2019.9  
 Work of FIRST TURBINE -----> .0  
 Work of SECOND TURBINE -----> 190480.8  
 TOTAL OUTPUT (including generator losses) --> 80258.4

(BASED ON LOWER HEATING VALUE)  
 Efficiency ---> 0.321420931316001370  
 Heat Rate ----> 10618.4746152843030

Table 2.2

\*\*\*\*\*  
 \*\* COMMAND MENU \*\*  
 \*\*\*\*\*

Please type the number or letter of the variable  
 you wish to change and PRESS ENTER.

- (TOGGLE VALUES)  
 1) Fuel used in power system  
     COALGAS  
 2) Method of finding products leaving gasifier  
     EXPERIMENTAL  
 -----
- (INPUT VALUES)  
 3) Temperature of air at inlet (Location 1)..... 298.15 K  
 4) Pressure of air at inlet (Location 1)..... 101.32 KPA  
 5) Mass flow rate of air at inlet (Location 1)... 287.13 KG/SEC  
 6) Fraction of air used to cool blades..... .050  
 7) Efficiency of air compressors..... .860  
 8) Efficiency of gas turbines..... .920  
 9) Efficiency of combustor..... .950  
 A) Efficiency of pipes..... .950  
 B) Efficiency of generator and drive..... .960  
 C) Efficiency of gasifier..... .960  
 D) Pressure in gasifier..... 2068.43 KPA  
 E) Fraction of air sent to gasifier..... .085  
 F) COAL STATISTICS  
     (Amounts in MASS FRACTIONS)  
         Carbon.... 81.300  
         Hydrogen... 5.300  
         Oxygen.... 9.800  
         Nitrogen... 1.700  
         Sulfur.... 1.900  
         Heating value of coal... 33725.0  
         Percent WATER in coal as received... .100  
         Percent ASH in coal as received... .100  
 G) Amount of water added per mole of dry coal.... .500  
 I) PRODUCT STATISTICS  
     (Amounts in moles - DRY & Sulfur FREE)  
         CO2.... .10300  
         CO..... .21200  
         H2..... .20100  
         CH4.... .04600  
         CnHm... .00300  
         N2..... .43500  
 J) AUXILIARY TURBINE STATUS  
     ---> THROTTLE <---



ENGLISH UNITS			
location	degrees F	psi	lbm/sec
1	76.67	14.69	633.00
2	76.58	14.68	633.00
3	719.52	171.80	633.00
4	719.52	163.21	633.00
5	719.52	163.21	53.80
6	719.52	155.05	53.80
7	1008.05	315.71	53.80
8	1008.05	299.92	53.80
9	1089.33	299.92	91.01
10	1089.33	284.93	91.01
11	1089.33	163.21	91.01
12	1089.33	155.05	91.01
13	719.52	163.21	547.54
14	719.52	155.05	547.54
15	2017.80	155.05	638.56
16	1616.27	70.84	638.56
17	1573.92	70.84	670.21
18	968.13	14.79	670.21
20	76.67	299.92	23.15
21	571.67	299.92	10.48
22	76.67	299.92	5.89
23	1340.00	299.92	2.31

METRIC UNITS			
location	degrees K	KPA	kg/sec
1	298.15	101.32	287.13
2	298.10	101.27	287.13
3	655.29	1184.81	287.13
4	655.29	1125.57	287.13
5	655.29	1125.57	24.41
6	655.29	1069.29	24.41
7	815.58	2177.29	24.41
8	815.58	2068.43	24.41
9	860.74	2068.43	41.28
10	860.74	1965.01	41.28
11	860.74	1125.57	41.28
12	860.74	1069.29	41.28
13	655.29	1125.57	248.37
14	655.29	1069.29	248.37
15	1376.56	1069.29	289.65
16	1153.48	488.57	289.65
17	1129.56	488.57	304.01
18	793.41	102.00	304.01
20	298.15	2068.43	10.50
21	573.15	2068.43	4.75
22	298.15	2068.43	2.67
23	1000.00	2068.43	1.05

WORK INPUT/OUTPUT (KW)  
 Work of FIRST COMPRESSOR -----> 106878.3  
 Work of SECOND COMPRESSOR -----> 4242.5  
 Work of FIRST TURBINE -----> .0  
 Work of SECOND TURBINE -----> 196625.5  
 TOTAL OUTPUT (including generator losses) --> 82084.6

HIGHER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS      7129.39        189.78  
 WET BASIS      6445.80        166.35

LOWER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS      6310.69        167.99  
 WET BASIS      5705.60        147.25

HEAT RATES & EFFICIENCIES  
 (HIGHER HEATING VALUES)  
 COAL EFFICIENCY -----> .2897  
 COAL HEAT RATE -----> 11779.5  
 GAS EFFICIENCY -----> .3085  
 GAS HEAT RATE -----> 11064.3  
 (LOWER HEATING VALUES)  
 COAL EFFICIENCY -----> .3000  
 COAL HEAT RATE -----> 11375.4  
 GAS EFFICIENCY -----> .3485  
 GAS HEAT RATE -----> 9793.8

RELATIVE FLOWS  
 LBM STEAM / LBM COAL -----> .4527  
 LBM STEAM&WATER / LBM COAL > .7073  
 LMB AIR / LBM COAL ----> 2.3243  
 66.1660 STD FT^3 GAS OUT / LBM COAL  
 .01221 LBM H2O / STD FT^3 GAS OUT  
 .47499 STD FT^3 AIR / STD FT^3 GAS

Table 2.3

\*\*\*\*\*  
 \*\* COMMAND MENU \*\*  
 \*\*\*\*\*

Please type the number or letter of the variable  
 you wish to change and PRESS ENTER.

- (TOGGLE VALUES)  
 1) Fuel used in power system  
     COALGAS  
 2) Method of finding products leaving gasifier  
     EXPERIMENTAL  
 -----
- (INPUT VALUES)  
 3) Temperature of air at inlet (Location 1)..... 298.15 K  
 4) Pressure of air at inlet (Location 1)..... 101.32 KPA  
 5) Mass flow rate of air at inlet (Location 1)... 287.13 KG/SEC  
 6) Fraction of air used to cool blades..... .050  
 7) Efficiency of air compressors..... .860  
 8) Efficiency of gas turbines..... .920  
 9) Efficiency of combustor..... .950  
 A) Efficiency of pipes..... .950  
 B) Efficiency of generator and drive..... .960  
 C) Efficiency of gasifier..... .960  
 D) Pressure in gasifier..... 2068.43 KPA  
 E) Fraction of air sent to gasifier..... .088  
 F) COAL STATISTICS  
     (Amounts in MASS FRACTIONS)  
         Carbon.... 81.300  
         Hydrogen... 5.300  
         Oxygen.... 9.800  
         Nitrogen... 1.700  
         Sulfur.... 1.900  
         Heating value of coal... 33725.0  
         Percent WATER in coal as received... .100  
         Percent ASH in coal as received... .100  
 G) Amount of water added per mole of dry coal.... .500  
 I) PRODUCT STATISTICS  
     (Amounts in moles - DRY & Sulfur FREE)  
         CO2.... .10300  
         CO.... .21200  
         H2.... .20100  
         CH4.... .04600  
         CnHm... .00300  
         N2.... .43500  
 J) AUXILIARY TURBINE STATUS  
     ---> AUXILIARY TURBINE <---

location	ENGLISH UNITS		
	degrees F	psi	lbm/sec
1	76.67	14.69	633.00
2	76.58	14.68	633.00
3	719.52	171.80	633.00
4	719.52	163.21	633.00
5	719.52	163.21	55.70
6	719.52	155.05	55.70
7	1008.05	315.71	55.70
8	1008.05	299.92	55.70
9	1089.33	299.92	94.22
10	1089.33	284.93	94.22
11	914.88	163.21	94.22
12	914.88	155.05	94.22
13	719.52	163.21	545.65
14	719.52	155.05	545.65
15	2024.98	155.05	639.87
16	1622.86	70.84	639.87
17	1580.28	70.84	671.52
18	973.39	14.79	671.52
20	76.67	299.92	23.97
21	571.67	299.92	10.85
22	76.67	299.92	6.10
23	1340.00	299.92	2.40

location	METRIC UNITS		
	degrees K	KPA	kg/sec
1	298.15	101.32	287.13
2	298.10	101.27	287.13
3	655.29	1184.81	287.13
4	655.29	1125.57	287.13
5	655.29	1125.57	25.27
6	655.29	1069.29	25.27
7	815.58	2177.29	25.27
8	815.58	2068.43	25.27
9	860.74	2068.43	42.74
10	860.74	1965.01	42.74
11	763.82	1125.57	42.74
12	763.82	1069.29	42.74
13	655.29	1125.57	247.51
14	655.29	1069.29	247.51
15	1380.54	1069.29	290.25
16	1157.14	488.57	290.25
17	1133.49	488.57	304.60
18	796.33	102.00	304.60
20	298.15	2068.43	10.87
21	573.15	2068.43	4.92
22	298.15	2068.43	2.77
23	1000.00	2068.43	1.09

WORK INPUT/OUTPUT (KW)  
 Work of FIRST COMPRESSOR -----> 106878.3  
 Work of SECOND COMPRESSOR -----> 4392.2  
 Work of FIRST TURBINE -----> 6372.2  
 Work of SECOND TURBINE -----> 197676.5  
 TOTAL OUTPUT (including generator losses) --> 89067.1

HIGHER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS      7129.39        189.78  
 WET BASIS      6445.80        166.35

LOWER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS      6310.69        167.99  
 WET BASIS      5705.60        147.25

HEAT RATES & EFFICIENCIES  
 (HIGHER HEATING VALUES)  
 COAL EFFICIENCY -----> .3037  
 COAL HEAT RATE -----> ~~14239.2~~  
 GAS EFFICIENCY -----> 3233  
 GAS HEAT RATE -----> ~~10556.8~~  
 (LOWER HEATING VALUES)  
 COAL EFFICIENCY -----> .3145  
 COAL HEAT RATE -----> ~~10853.6~~  
 GAS EFFICIENCY -----> .3652  
 GAS HEAT RATE -----> ~~9344.5~~

RELATIVE FLOWS  
 LBM STEAM / LBM COAL -----> .4527  
 LBM STEAM&WATER / LBM COAL > .7073  
 LMB AIR / LBM COAL -----> 2.3243  
 66.1660 STD FT^3 GAS OUT / LBM COAL  
 .01221 LBM H2O / STD FT^3 GAS OUT  
 .47499 STD FT^3 AIR / STD FT^3 GAS

Table 2.4

\*\*\*\*\*  
\*\* COMMAND MENU \*\*  
\*\*\*\*\*

Please type the number or letter of the variable  
you wish to change and PRESS ENTER.

- (TOGGLE VALUES)  
1) Fuel used in power system  
COALGAS  
2) Method of finding products leaving gasifier  
THEORETICAL -> (EQUILIBRIUM)  
-----  
(INPUT VALUES)  
3) Temperature of air at inlet (Location 1)..... 298.15 K  
4) Pressure of air at inlet (Location 1)..... 101.32 KPA  
5) Mass flow rate of air at inlet (Location 1)... 287.13 KG/SEC  
6) Fraction of air used to cool blades..... .050  
7) Efficiency of air compressors..... .860  
8) Efficiency of gas turbines..... .920  
9) Efficiency of combustor..... .950  
A) Efficiency of pipes..... .950  
B) Efficiency of generator and drive..... .960  
C) Efficiency of gasifier..... .960  
D) Pressure in gasifier..... 2068.43 KPA  
E) Fraction of air sent to gasifier..... .085  
F) COAL STATISTICS  
(Amounts in MASS FRACTIONS)  
Carbon.... 81.300  
Hydrogen... 5.300  
Oxygen.... 9.300  
Nitrogen... 1.700  
Sulfur.... 1.900  
Heating value of coal... 33725.0  
Percent WATER in coal as received... .100  
Percent ASH in coal as received... .100  
G) Amount of water added per mole of dry coal.... .500

GASIFIER PRODUCTS

CO2 -----> .2659543  
 CO -----> .2940301  
 H2 -----> .3370807  
 CH4 -----> .1290224  
 C2H6 -----> .0000098  
 N2 -----> .8167270  
 H2O -----> .2370947  
 H2S -----> .0060322

location	ENGLISH UNITS		
	degrees F	psi	lbm/sec
1	76.67	14.69	633.00
2	76.58	14.68	633.00
3	719.52	171.80	633.00
4	719.52	163.21	633.00
5	719.52	163.21	53.80
6	719.52	155.05	53.80
7	1008.05	315.71	53.80
8	1008.05	299.92	53.80
9	1391.71	299.92	91.01
10	1391.71	284.93	91.01
11	1391.71	163.21	91.01
12	1391.71	155.05	91.01
13	719.52	163.21	547.54
14	719.52	155.05	547.54
15	2013.36	155.05	638.56
16	1612.49	70.84	638.56
17	1570.32	70.84	670.21
18	965.52	14.79	670.21
20	76.67	299.92	23.15
21	571.67	299.92	10.48
22	76.67	299.92	5.89
23	1340.00	299.92	2.31

location	METRIC UNITS		
	degrees K	KPA	kg/sec
1	298.15	101.32	287.13
2	298.10	101.27	287.13
3	655.29	1184.81	287.13
4	655.29	1125.57	287.13
5	655.29	1125.57	24.41
6	655.29	1069.29	24.41
7	815.58	2177.29	24.41
8	815.58	2068.43	24.41
9	1028.73	2068.43	41.28
10	1028.73	1965.01	41.28
11	1028.73	1125.57	41.28
12	1028.73	1069.29	41.28
13	655.29	1125.57	248.37
14	655.29	1069.29	248.37
15	1374.09	1069.29	289.65
16	1151.38	488.57	289.65
17	1127.96	488.57	304.01
18	791.96	102.00	304.01
20	298.15	2068.43	10.50
21	573.15	2068.43	4.75
22	298.15	2068.43	2.67
23	1000.00	2068.43	1.05

WORK INPUT/OUTPUT (KW)  
 Work of FIRST COMPRESSOR -----> 106878.3  
 Work of SECOND COMPRESSOR -----> 4242.5  
 Work of FIRST TURBINE -----> .0  
 Work of SECOND TURBINE -----> 196258.1  
 TOTAL OUTPUT (including generator losses) --> 81731.8

HIGHER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS      6734.40        185.53  
 WET BASIS      6159.65        164.44

LOWER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS      5928.52        163.33  
 WET BASIS      5422.55        144.76

HEAT RATES & EFFICIENCIES  
 (HIGHER HEATING VALUES)  
 COAL EFFICIENCY -----> .2885  
 COAL HEAT RATE -----> 11829.9  
 GAS EFFICIENCY -----> .3214  
 GAS HEAT RATE -----> 10618.7  
 (LOWER HEATING VALUES)  
 COAL EFFICIENCY -----> .2988  
 COAL HEAT RATE -----> 11424.0  
 GAS EFFICIENCY -----> .3651  
 GAS HEAT RATE -----> 9348.0

RELATIVE FLOWS  
 LBM STEAM / LBM COAL -----> .4527  
 LBM STEAM&WATER / LBM COAL > .7073  
 LMB AIR / LBM COAL ----> 2.3244  
 64.2324 STD FT^3 GAS OUT / LBM COAL  
 .01258 LBM H2O / STD FT^3 GAS OUT  
 .48931 STD FT^3 AIR / STD FT^3 GAS



Table 2.5

\*\*\*\*\*  
 \*\* COMMAND MENU \*\*  
 \*\*\*\*\*

Please type the number or letter of the variable  
 you wish to change and PRESS ENTER.

- 
- (TOGGLE VALUES)
- 1) Fuel used in power system  
COALGAS
  - 2) Method of finding products leaving gasifier  
THEORETICAL -> (EQUILIBRIUM)
- 
- (INPUT VALUES)
- 3) Temperature of air at inlet (Location 1)..... 298.15 K
  - 4) Pressure of air at inlet (Location 1)..... 101.32 KPA
  - 5) Mass flow rate of air at inlet (Location 1)... 287.13 KG/SEC
  - 6) Fraction of air used to cool blades..... .050
  - 7) Efficiency of air compressors..... .860
  - 8) Efficiency of gas turbines..... .920
  - 9) Efficiency of combustor..... .950
  - A) Efficiency of pipes..... .950
  - B) Efficiency of generator and drive..... .960
  - C) Efficiency of gasifier..... .960
  - D) Pressure in gasifier..... 2068.43 KPA
  - E) Fraction of air sent to gasifier..... .088
- F) COAL STATISTICS
- (Amounts in MASS FRACTIONS)
- Carbon.... 81.300
  - Hydrogen... 5.300
  - Oxygen.... 9.800
  - Nitrogen... 1.700
  - Sulfur.... 1.900
  - Heating value of coal... 33725.0
  - Percent WATER in coal as received... .100
  - Percent ASH in coal as received... .100
- G) Amount of water added per mole of dry coal.... .500

GASIFIER PRODUCTS

CO2 -----> .2659543  
 CO -----> .2940301  
 H2 -----> .3370807  
 CH4 -----> .1290224  
 C2H6 -----> .0000098  
 N2 -----> .8167270  
 H2O -----> .2370947  
 H2S -----> .0060322

ENGLISH UNITS

location	degrees F	psi	lbm/sec
1	76.67	14.69	633.00
2	76.58	14.68	633.00
3	719.52	171.80	633.00
4	719.52	163.21	633.00
5	719.52	163.21	55.70
6	719.52	155.05	55.70
7	1008.05	315.71	55.70
8	1008.05	299.92	55.70
9	1391.71	299.92	94.22
10	1391.71	284.93	94.22
11	1190.21	163.21	94.22
12	1190.21	155.05	94.22
13	719.52	163.21	545.65
14	719.52	155.05	545.65
15	2014.61	155.05	639.87
16	1614.10	70.84	639.87
17	1571.94	70.84	671.52
18	967.44	14.79	671.52
20	76.67	299.92	23.97
21	571.67	299.92	10.85
22	76.67	299.92	6.10
23	1340.00	299.92	2.40

METRIC UNITS

location	degrees K	KPA	kg/sec
1	298.15	101.32	287.13
2	298.10	101.27	287.13
3	655.29	1184.81	287.13
4	655.29	1125.57	287.13
5	655.29	1125.57	25.27
6	655.29	1069.29	25.27
7	815.58	2177.29	25.27
8	815.58	2068.43	25.27
9	1028.73	2068.43	42.74
10	1028.73	1965.01	42.74
11	916.78	1125.57	42.74
12	916.78	1069.29	42.74
13	655.29	1125.57	247.51
14	655.29	1069.29	247.51
15	1374.78	1069.29	290.24
16	1152.28	488.57	290.24
17	1128.85	488.57	304.60
18	793.02	102.00	304.60
20	298.15	2068.43	10.87
21	573.15	2068.43	4.92
22	298.15	2068.43	2.77
23	1000.00	2068.43	1.09

WORK INPUT/OUTPUT (KW)  
 Work of FIRST COMPRESSOR -----> 106878.3  
 Work of SECOND COMPRESSOR -----> 4392.2  
 Work of FIRST TURBINE -----> 7377.8  
 Work of SECOND TURBINE -----> 196842.9  
 TOTAL OUTPUT (including generator losses) --> 89232.2

HIGHER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS        6734.40            185.53  
 WET BASIS        6159.65            164.44

LOWER HEATING VALUES  
 KJ/KG            BTU/SCF  
 DRY BASIS        5928.52            163.33  
 WET BASIS        5422.55            144.76

HEAT RATES & EFFICIENCIES  
 (HIGHER HEATING VALUES)  
 COAL EFFICIENCY -----> .3042  
 COAL HEAT RATE -----> 11217.9  
 GAS EFFICIENCY -----> .3389  
 GAS HEAT RATE -----> 10069.4  
 (LOWER HEATING VALUES)  
 COAL EFFICIENCY -----> .3151  
 COAL HEAT RATE -----> 10833.1  
 GAS EFFICIENCY -----> .3850  
 GAS HEAT RATE -----> 8864.4

RELATIVE FLOWS  
 LBM STEAM / LBM COAL ----> .4527  
 LBM STEAM&WATER / LBM COAL > .7073  
 LMB AIR / LBM COAL ----> 2.3244  
 64.2324    STD FT^3 GAS OUT / LBM COAL  
 .01258    LBM H2O / STD FT^3 GAS OUT  
 .48931    STD FT^3 AIR / STD FT^3 GAS

Table 2.6 IGCC System Computer Model Sample Run

\*\*\*\*\*  
 \*\* COMMAND MENU \*\*  
 \*\*\*\*\*

Please type the number or letter of the variable  
 you wish to change and PRESS ENTER.

-----  
 (TOGGLE VALUES)

- 1) Fuel used in power system  
     COALGAS
- 2) Method of finding products leaving gasifier  
     THEORETICAL -> (EQUILIBRIUM)

-----  
 (INPUT VALUES)

- 3) Temperature of air at inlet (Location 1)..... 298.15 K
- 4) Pressure of air at inlet (Location 1)..... 101.32 KPA
- 5) Mass flow rate of air at inlet (Location 1)... 287.13 KG/SEC
- 6) Fraction of air used to cool blades..... .050
- 7) Efficiency of air compressors..... .860
- 8) Efficiency of gas turbines..... .920
- 9) Efficiency of combustor..... .950
- A) Efficiency of pipes..... .950
- B) Efficiency of generator and drive..... .960
- C) Efficiency of gasifier..... .960
- D) Pressure in gasifier..... 2068.43 KPA
- E) Fraction of air sent to gasifier..... .085
- F) COAL STATISTICS  
     (Amounts in MASS FRACTIONS)  
         Carbon.... 81.300  
         Hydrogen... 5.300  
         Oxygen.... 9.800  
         Nitrogen... 1.700  
         Sulfur.... 1.900  
         Heating value of coal... 33725.0  
         Percent WATER in coal as received... .100  
         Percent ASH in coal as received... .100
- G) Amount of water added per mole of dry coal.... .500
- H) Amount of air added per mole of dry coal..... .215

- J) AUXILIARY TURBINE STATUS  
     ---> AUXILIARY TURBINE <---

-----> INPUT SELECTION <-----

(ENTER -> 0 <- TO CONTINUE ONWARD)

GASIFIER PRODUCTS

CO2 -----> .3129231  
 CO -----> .2153383  
 H2 -----> .2888830  
 CH4 -----> .1607398  
 C2H6 -----> .0000127  
 N2 -----> .8167270  
 H2O -----> .1511596  
 H2S -----> .0060322

ENGLISH UNITS

METRIC

UNITS

POINT	DEGREES F	PSI	LBM/SEC	DEGREES K	KPA	KG/SEC
1	76.67	14.69	607.04	298.15	101.32	275.35
2	76.58	14.68	607.04	298.10	101.27	275.35
3	812.16	192.41	607.04	706.76	1326.98	275.35
4	812.16	182.79	607.04	706.76	1260.63	275.35
5	812.16	182.79	51.60	706.76	1260.63	23.41
6	812.16	173.65	51.60	706.76	1197.60	23.41
7	1067.99	315.71	51.60	848.88	2177.29	23.41
8	1067.99	299.92	51.60	848.88	2068.43	23.41
9	1311.50	299.92	87.28	984.17	2068.43	39.59
10	1311.50	284.93	87.28	984.17	1965.01	39.59
11	1160.88	182.79	87.28	900.49	1260.63	39.59
12	1160.88	173.65	87.28	900.49	1197.60	39.59
13	812.16	182.79	525.09	706.76	1260.63	238.18
14	812.16	173.65	525.09	706.76	1197.60	238.18
15	2037.61	173.65	612.37	1387.56	1197.60	277.77
16	1615.06	76.40	612.37	1152.81	526.92	277.77
17	1577.14	76.40	642.72	1131.75	526.92	291.54
18	946.16	14.79	642.72	781.20	102.00	291.54
19	1249.64	14.79	646.32	949.80	102.00	293.17
20	76.67	299.92	22.20	298.15	2068.43	10.07
21	571.67	299.92	10.05	573.15	2068.43	4.56
22	76.67	299.92	5.65	298.15	2068.43	2.56
23	1340.00	299.92	2.22	1000.00	2068.43	1.01
24	1043.10	14.79	642.72	835.06	102.00	291.54
25	423.05	14.79	642.72	490.58	102.00	291.54
26	370.77	14.79	642.72	461.54	102.00	291.54
27	114.13	1.45	98.67	318.96	10.00	44.76
28	114.22	20.30	107.95	319.01	140.00	48.97
29	228.43	20.30	120.85	382.46	140.00	54.82

30	233.08	910.02	111.50		385.04 6276.00	50.58
31	533.13	910.02	111.50		551.74 6276.00	50.58
32	1049.66	910.02	108.15		838.70 6276.00	49.06
33	1044.26	864.52	108.15		835.70 5962.20	49.06
34	810.61	314.94	108.15		705.90 2172.00	49.06
35	810.61	314.94	98.10		705.90 2172.00	44.50
36	678.89	166.26	98.10		632.72 1146.65	44.50
37	652.07	166.26	107.17		617.82 1146.65	48.61
38	231.18	21.37	107.17		383.99 147.37	48.61
39	228.43	20.30	12.90		382.46 140.00	5.85
40	231.18	21.37	94.27		383.99 147.37	42.76
41	114.13	1.45	94.27		318.96 10.00	42.76
42	114.13	1.45	9.28		318.96 10.00	4.21
43	114.13	1.45	107.95		318.96 10.00	48.97
44	229.24	175.01	9.35		382.91 1207.00	4.24
45	370.52	175.01	9.35		461.40 1207.00	4.24
46	370.52	175.01	9.07		461.40 1207.00	4.11
47	366.37	166.26	9.07		459.09 1146.65	4.11
48	533.13	910.02	3.34		551.74 6276.00	1.52
49	370.52	175.01	.28		461.40 1207.00	.13
50	76.67	14.69	4.12		298.15 101.32	1.87
51	1249.64	14.79	.51		949.80 102.00	.23

TURBINE PRODUCTS

N2	----->	8.85554941
O2	----->	1.54963079
CO2	----->	.68902643
CO	----->	.00000000
H2O	----->	.76759235
H2	----->	.00000000
CH4	----->	.00000000
C2H6	----->	.00000000
SO2	----->	.00603224
H2S	----->	.00000000
OH	----->	.00000000
NO	----->	.00000000

WORK INPUT/OUTPUT (KW)

WORK FROM MAIN GAS TURBINE (GE MS7001 HEAVY DUTY)

Work of FIRST COMPRESSOR	----->	-117956.9
Work of SECOND COMPRESSOR	----->	-3621.5
Work of FIRST TURBINE	----->	5065.8
Work of SECOND TURBINE (optional)	->	196727.5

TOTAL OUTPUT OF GAS TURBINE (GE MS7001)--> 77006.3  
Net Work of RECOVERY TURBINES -----> 49874.5  
\*\*\*\*\* TOTAL OUTPUT OF SYSTEM \*\*\*\*\* -> 126880.8

HIGHER HEATING VALUES

	KJ/KG	BTU/SCF
DRY BASIS	6442.68	183.34
WET BASIS	6082.97	169.14

LOWER HEATING VALUES

	KJ/KG	BTU/SCF
DRY BASIS	5709.15	162.46
WET BASIS	5390.40	149.88

HEAT RATES & EFFICIENCIES

GAS TURBINE EFFICIENCIES

(HIGHER HEATING VALUES)

COAL EFFICIENCY -----> .2835  
COAL HEAT RATE -----> 12040.9  
GAS EFFICIENCY -----> .3198  
GAS HEAT RATE -----> 10673.5

(LOWER HEATING VALUES)

COAL EFFICIENCY -----> .2935  
COAL HEAT RATE -----> 11627.8  
GAS EFFICIENCY -----> .3608  
GAS HEAT RATE -----> 9458.3

COMBINED CYCLE EFFICIENCY

(HIGHER HEATING VALUES)

COAL EFFICIENCY -----> .3978  
COAL HEAT RATE -----> 8578.7

(LOWER HEATING VALUES)

COAL EFFICIENCY -----> .4120  
COAL HEAT RATE -----> 8284.4

RELATIVE FLOWS

LBM STEAM / LBM COAL ----> .4527  
LBM STEAM&WATER / LBM COAL > .7073  
LMB AIR / LBM COAL ----> 2.3244  
60.1027 STD FT^3 GAS OUT / LBM COAL  
.01178 LBM H2O / STD FT^3 GAS OUT  
.52293 STD FT^3 AIR / STD FT^3 GAS

## 2.7 Parametric Analysis

There are several parameters in the model (such as the compressor and turbine small stage efficiencies) for which exact numerical values are unavailable. Also, some parameters such as the percent moisture in the coal change on a day to day basis. Therefore, parametric analysis were performed to determine the effects of changes in several of the design variables on the overall system performance. Gas turbine stage efficiency, booster compressor stage efficiency, gasifier pressure, percent moisture in coal, and percent ash in coal were varied over a reasonable range. The results of the parametric analyses are discussed below with the following coal as input:

Table 2.7

COAL STATISTICS	
(Amounts in MASS FRACTIONS)	
Carbon....	81.300
Hydrogen...	5.300
Oxygen....	9.800
Nitrogen...	1.700
Sulfur....	1.900
Heating value of coal...	33725.0

### 2.7.1 Turbine Isentropic Efficiency

The small stage efficiency of the gas turbine was found to have a considerable effect on the exhaust temperatures and the turbine output power. Although this parameter is somewhat given for an existing turbine, it is easy to see how a high efficiency turbine is of the utmost importance. Figure 2.2 illustrates how the temperatures at locations 16, 17, and 18 are changed over a range of turbine efficiency. As the efficiency of the gas turbine increases, all three of these temperatures decrease. This is due to the fact that more energy is converted into usable work. Figure 2.3 shows how the turbine output sharply increases as the turbine efficiency increases. Although a certain amount of work can be regained at later steam turbine sections of the IGCC plant, the net affect of a decrease in turbine efficiency is still a loss in the overall power plant efficiency.



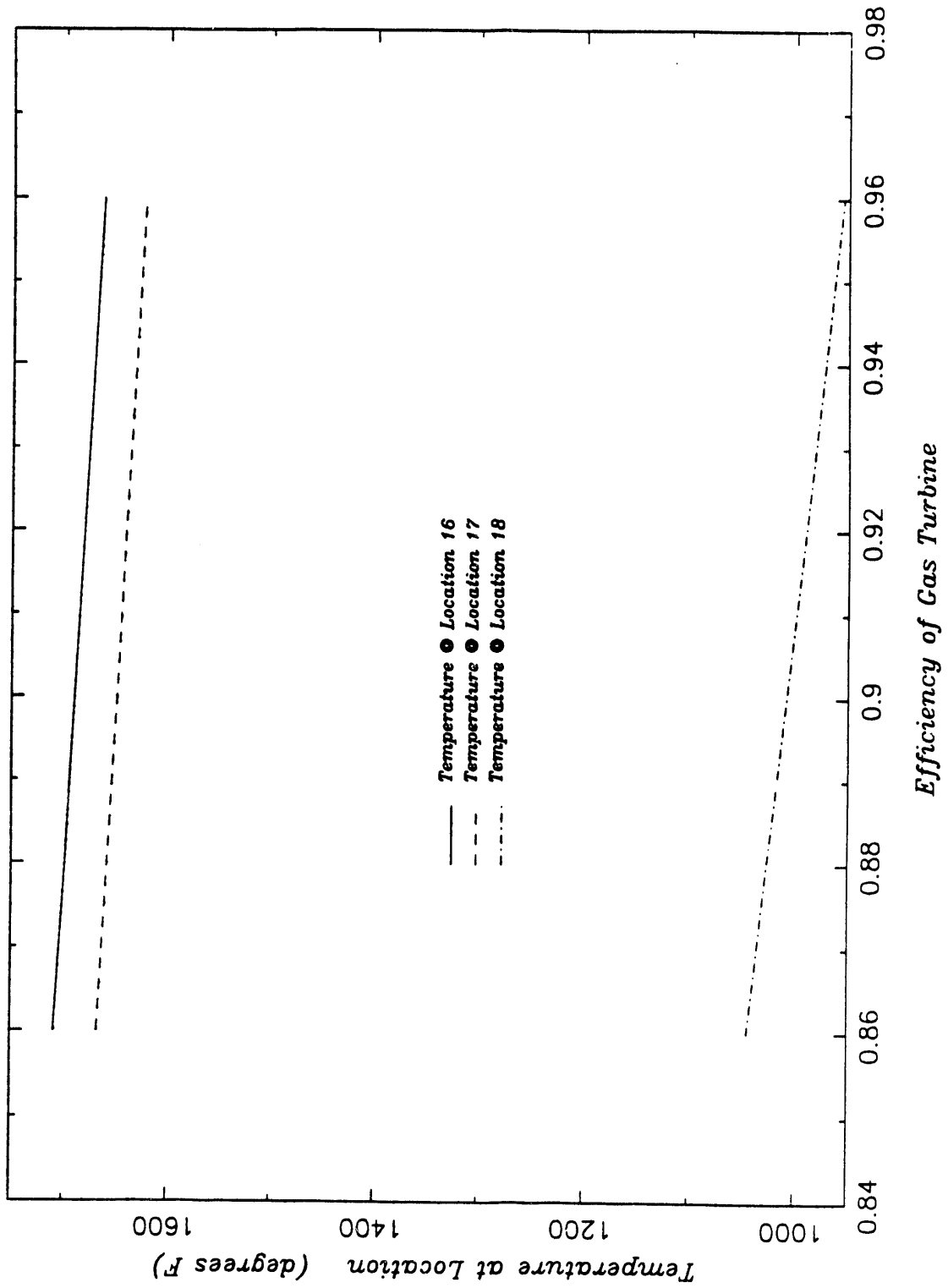


Figure 2.2 Temperature verses Efficiency of Gas Turbine

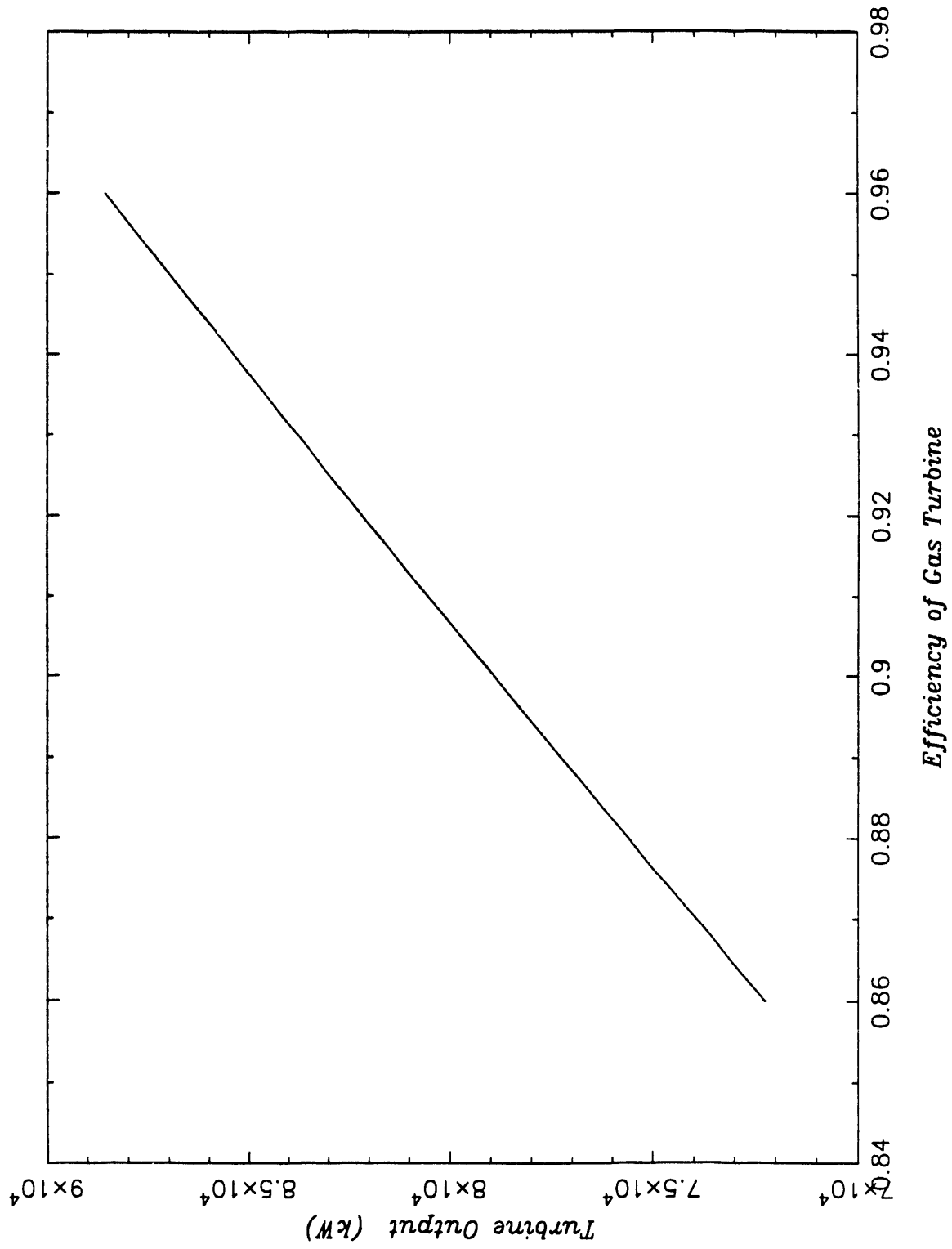


Figure 2.3 Turbine Output versus Efficiency of Gas Turbine

### **2.7.2 Booster Compressor Isentropic Efficiency**

A change in the efficiency of the booster compressor altered the temperature after the compressor, which also changed the gasifier outlet temperature. Figure 2.4 illustrates the effect of these two temperatures as the efficiency of the compressor increases. However, Figure 2.5 shows that the turbine output is only slightly changed over a wide range of booster compressor efficiency.

### **2.7.3 Pressure in Gasifier**

The pressure within the gasifier was found to greatly influence the temperatures before and after the gasifier as well as turbine output. Figure 2.6 illustrates the temperature changes around the gasifier. The temperature before the gasifier becomes higher at higher gasifier pressures as a result of the work input at the booster compressor. Consequently, the product gas temperature also increases in as the pressure in the gasifier is increased. Figure 2.7 shows that the turbine output slightly increases as the gasifier pressure increases.

### **2.7.4 Percent Water in Inlet Coal**

The percent water in the inlet coal greatly affected gasifier temperatures, needed inlet coal mass flow rates, and of course turbine output work. Figure 2.8 shows how the gasifier exit temperature drops with an increase in percent water in the coal. In fact, Figure 2.9 shows that the temperature had even dropped although the coal mass flow rate had increased. Figure 2.10 illustrates how the turbine output decreases with an increase in the amount of moisture in the coal.

### **2.7.5 Percent Ash in Inlet Coal**

The percent ash in the coal was found to have effects similar to the percent water in the coal. Figure 2.11 illustrates how the gasifier exit temperature slightly decreases as the

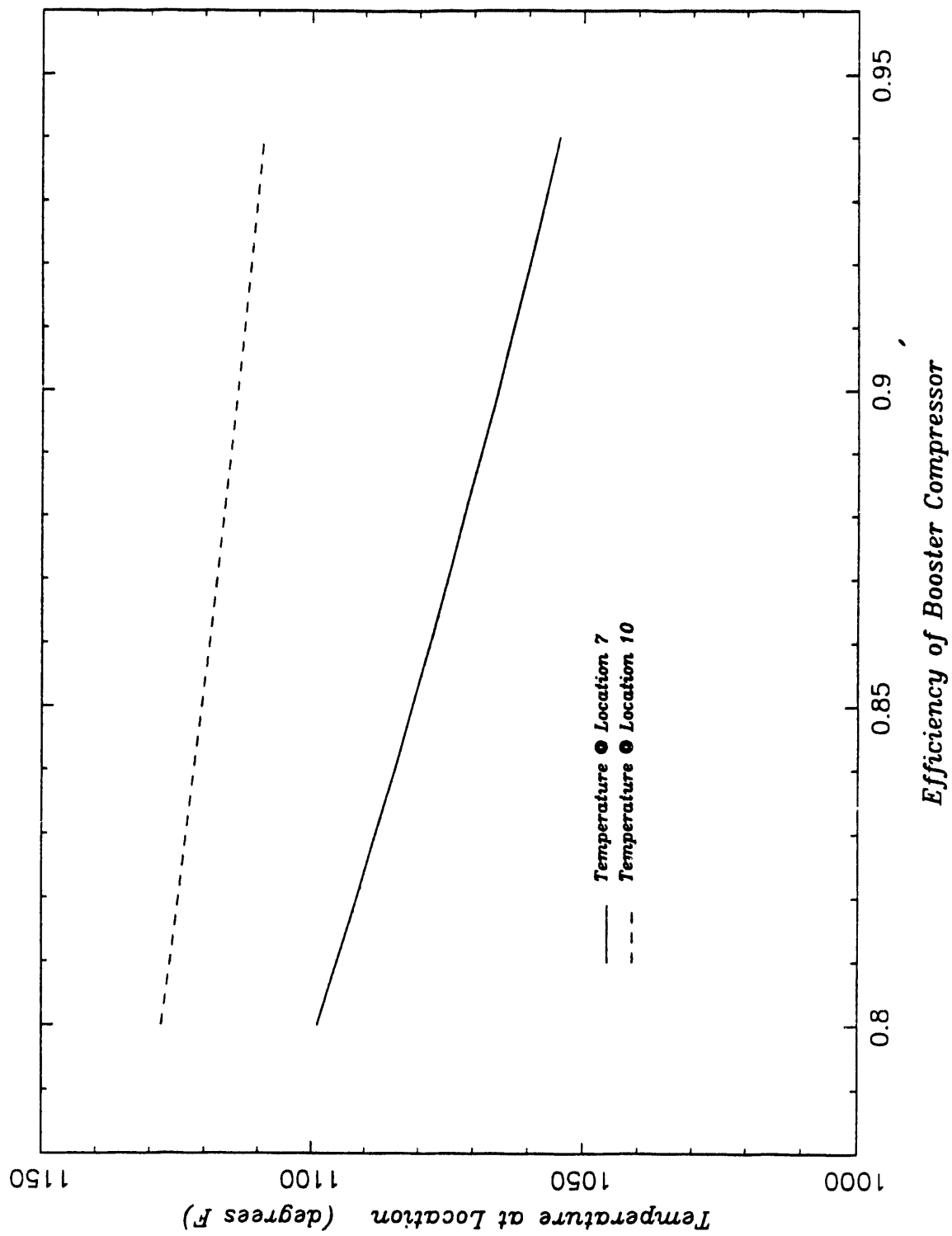


Figure 2.4 Temperature verses Efficiency of Booster Compressor

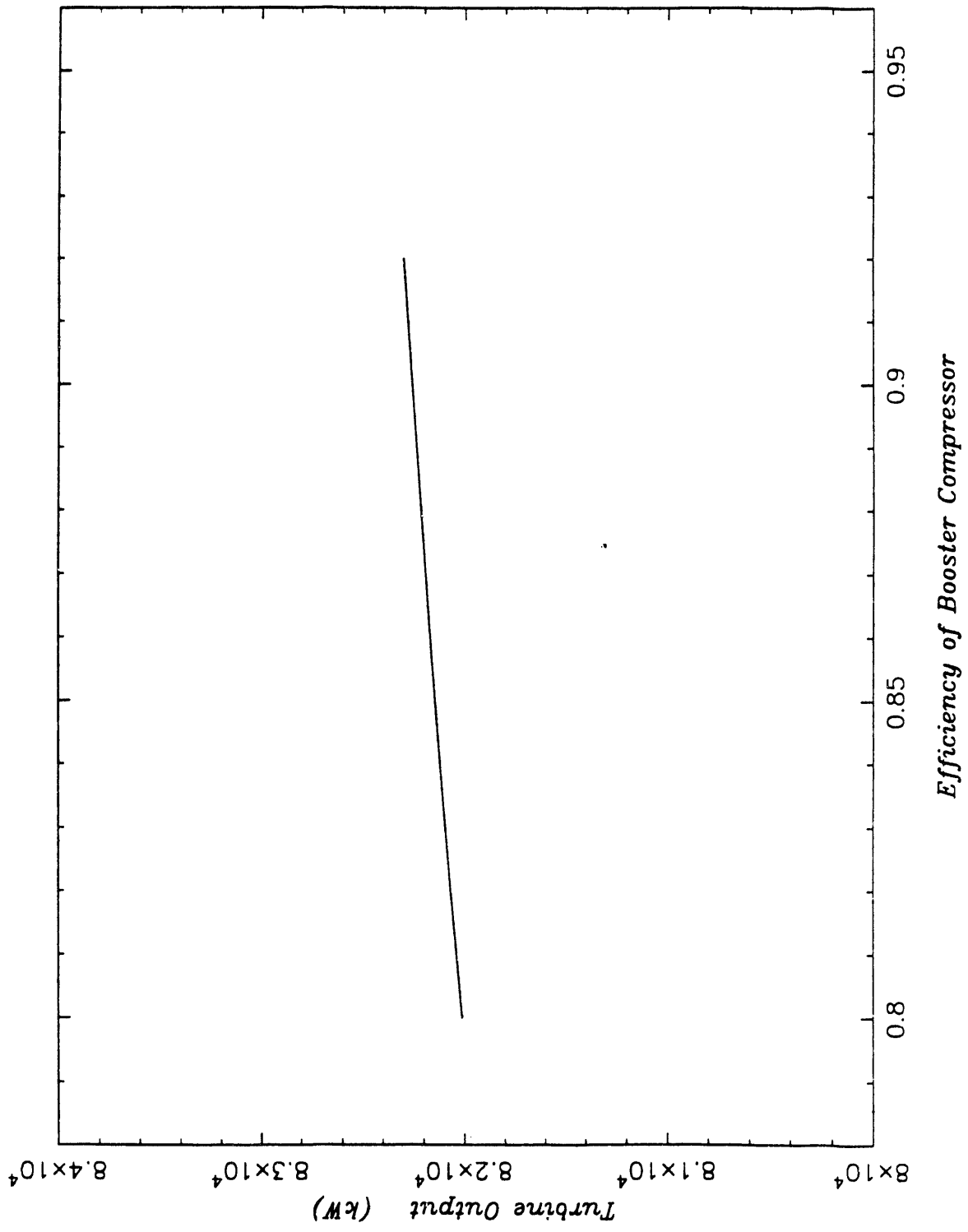


Figure 2.5 Turbine Output versus Efficiency of Booster Compressor

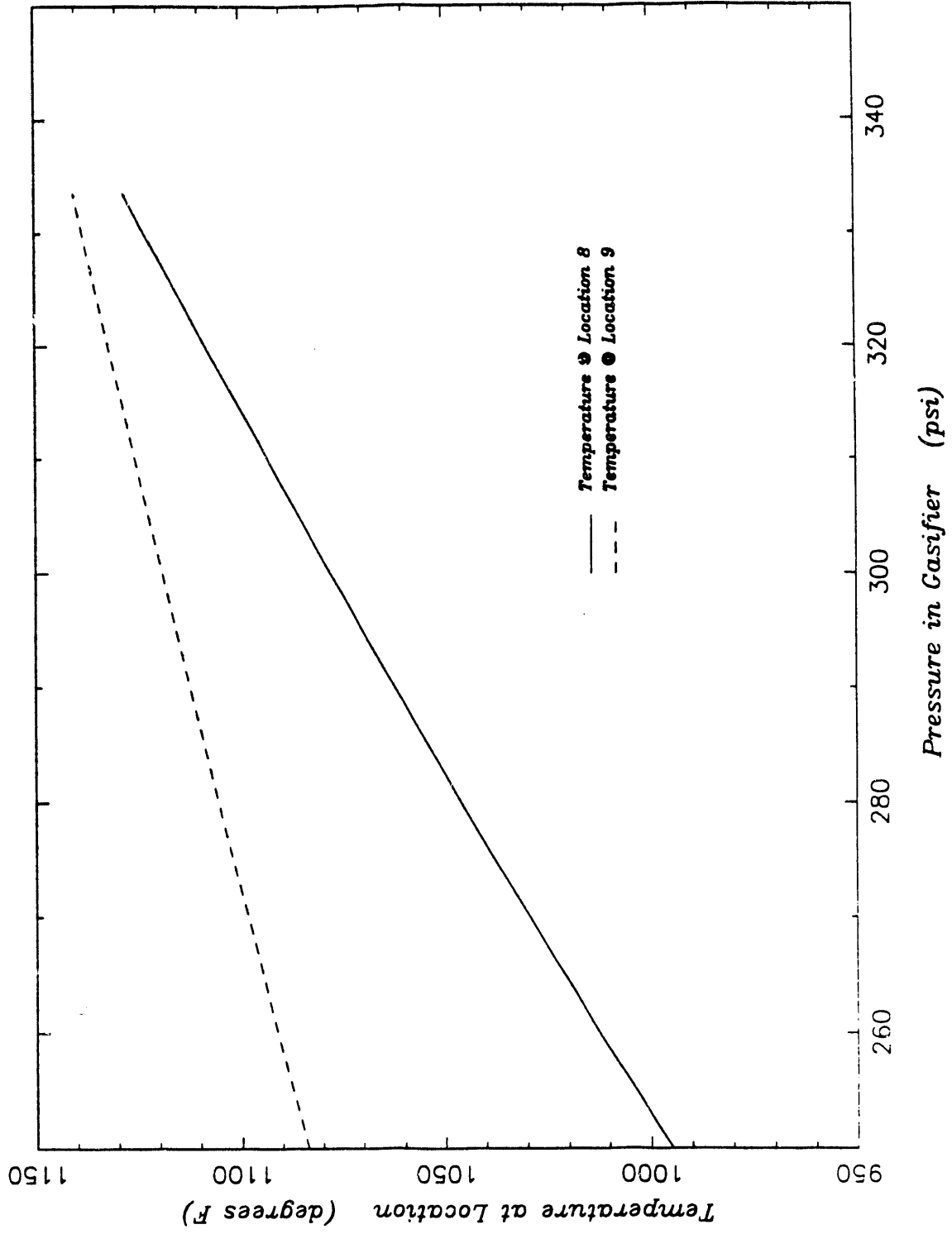


Figure 2.6 Temperature verses Pressure in Gasifier

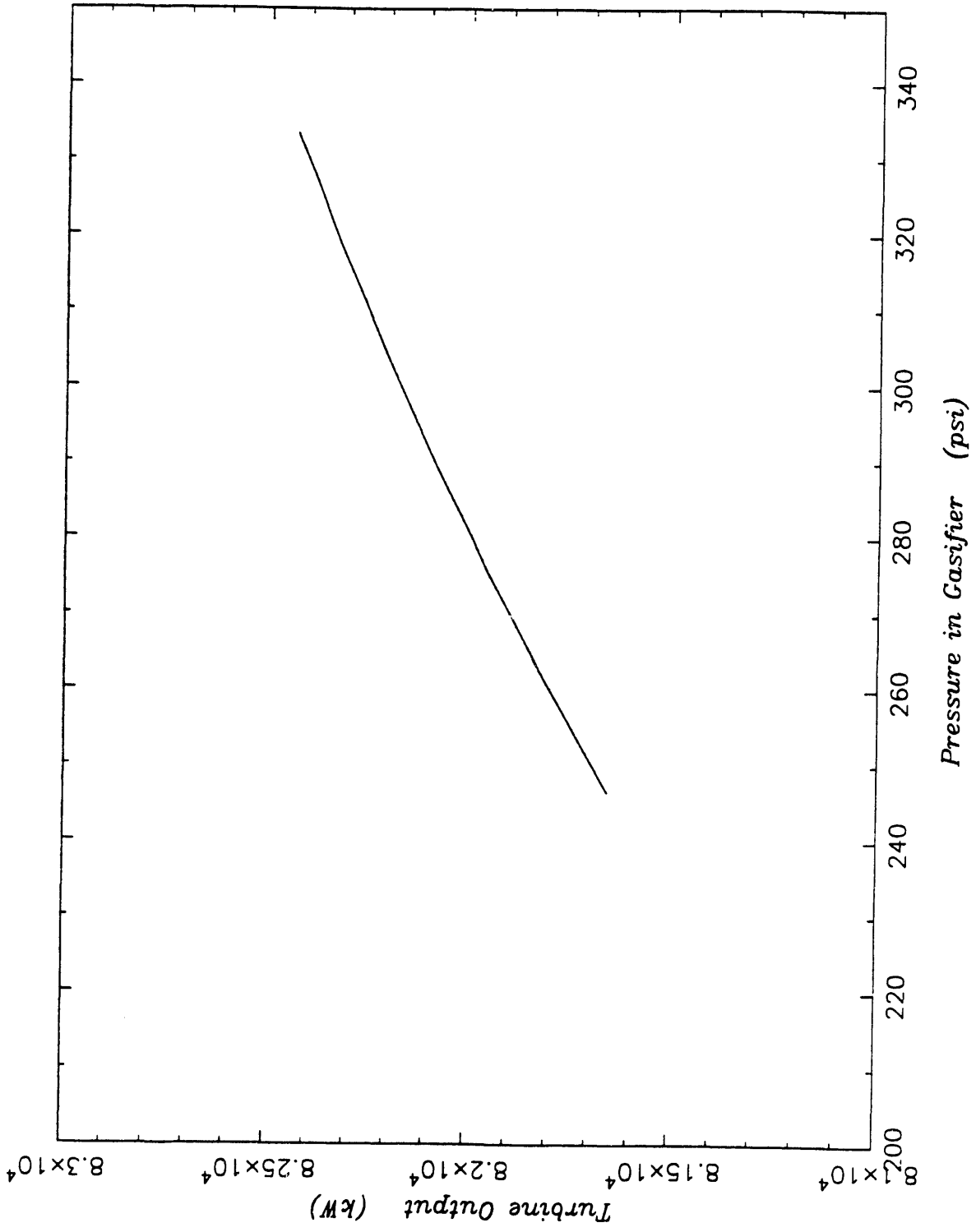


Figure 2.7 Turbine Output versus Pressure in Gasifier

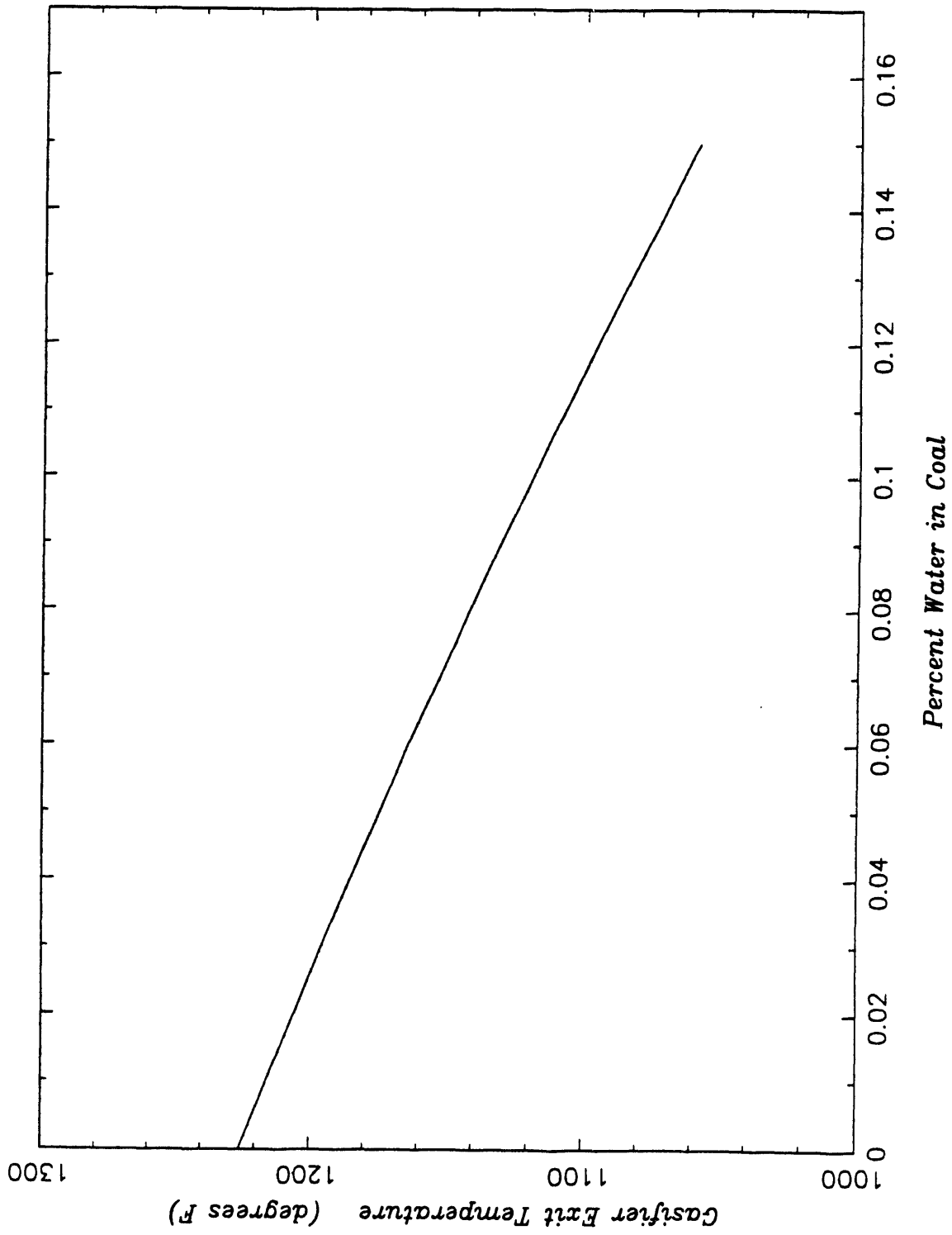


Figure 2.8 Gasifier Exit Temperature versus Percent Water in Coal



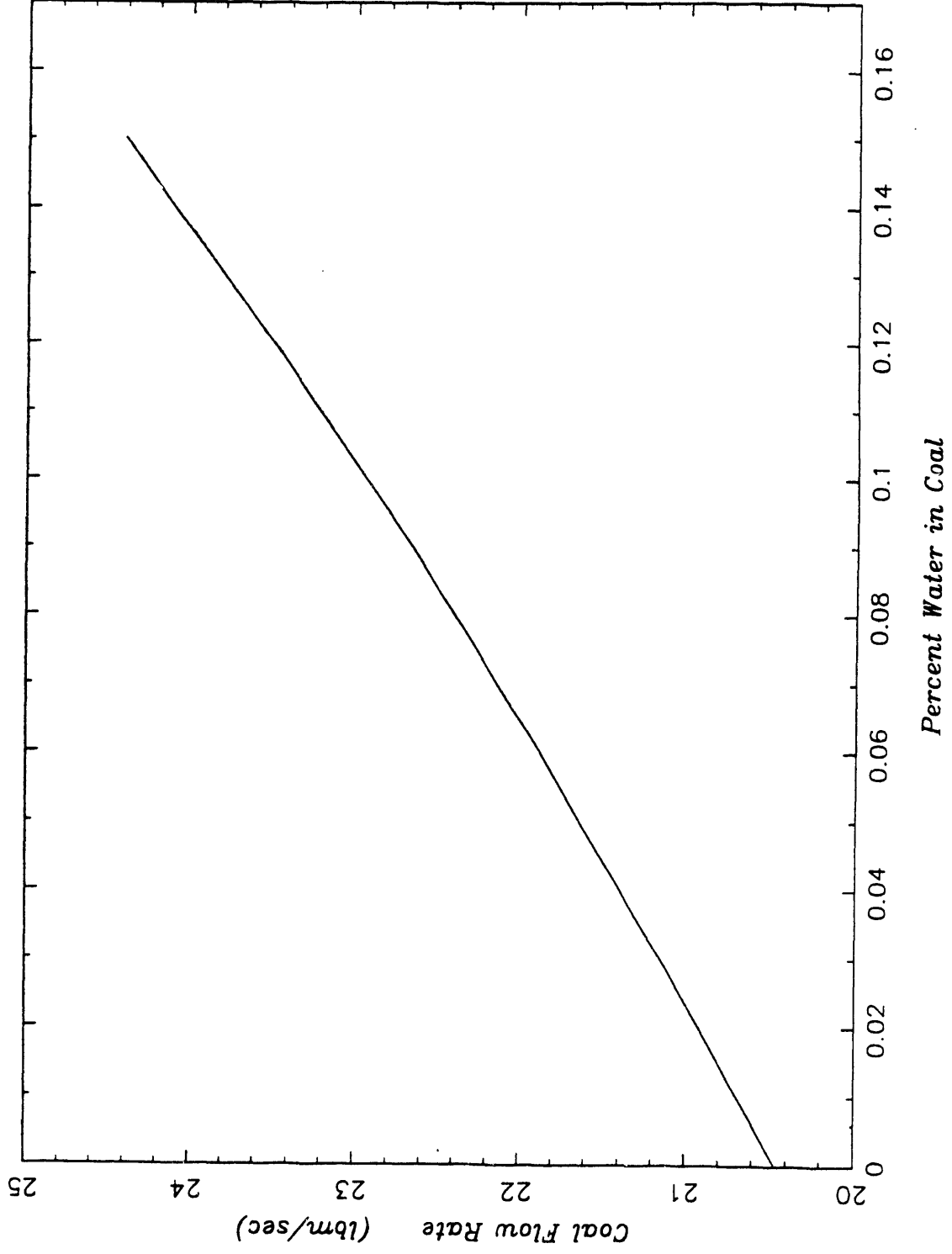


Figure 2.9 Coal Flow Rate versus Percent Water in Coal

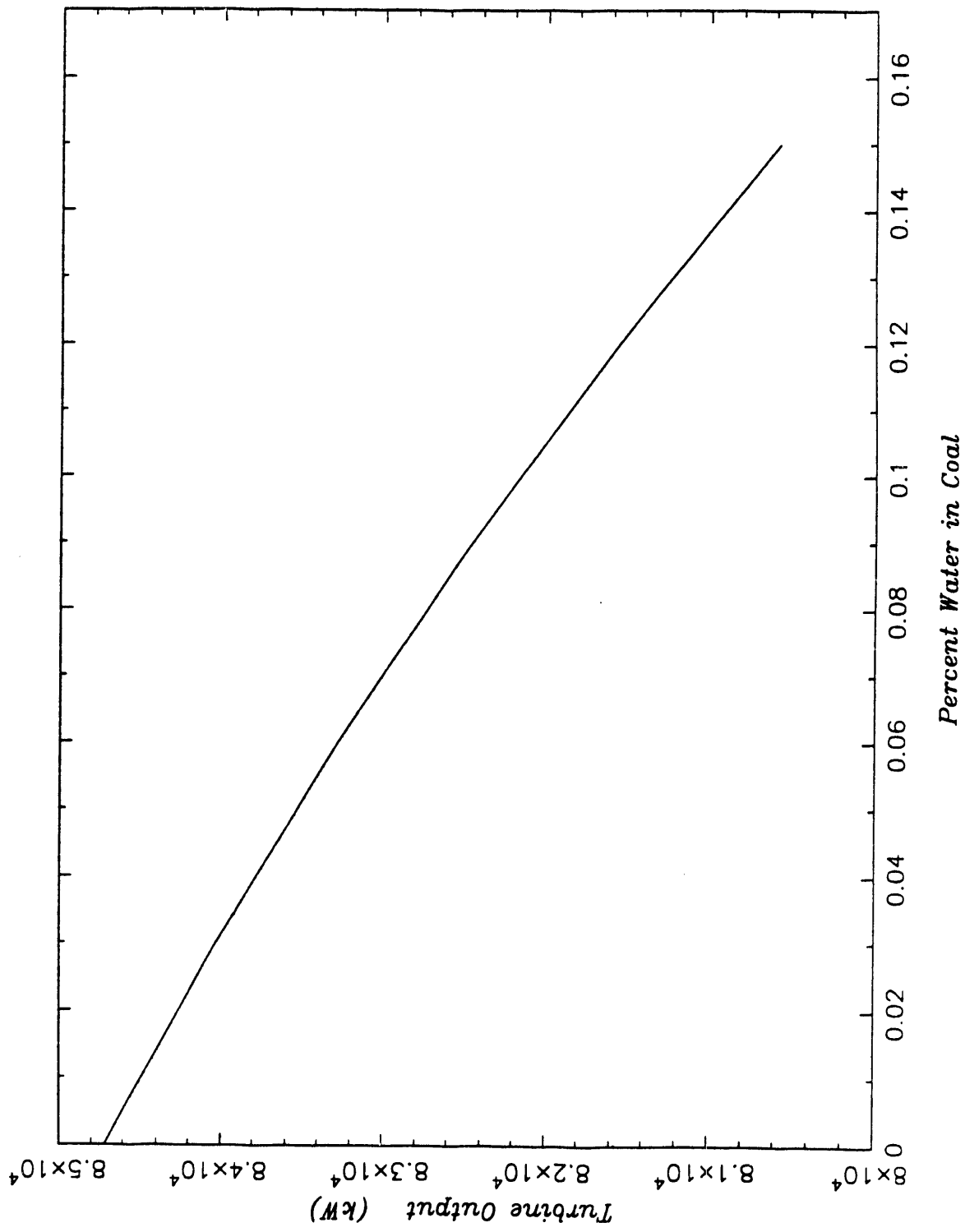


Figure 2.10 Turbine Output verses Percent Water in Coal

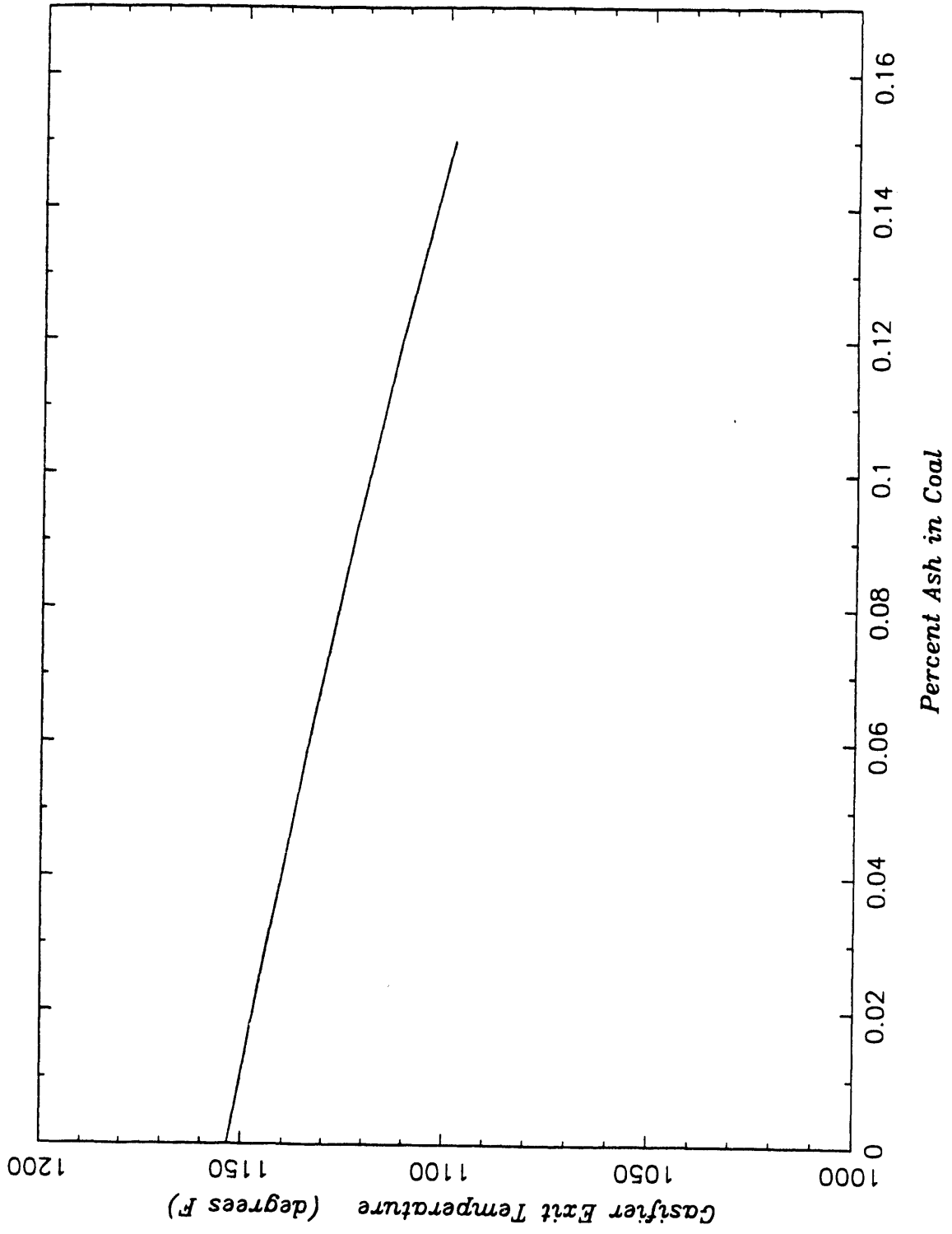


Figure 2.11 Gasifier Exit Temperature versus Percent Ash in Inlet Coal

percent ash is increased. Energy was lost in this system when the hot ash left without contributing in a positive way. Figure 2.12 illustrates the increase in the feed of the coal. Because the computer model was designed to work with a certain prescribed turbine output, more coal was needed when there were large losses due to the percent water or percent ash in the coal. Figure 2.13 shows how the ash flow rate increases as the percent ash in the coal becomes larger. It should not be a surprise to find that this relation is almost linear. Even though more coal was added to the system when the percent ash increased, there still existed turbine output losses. Figure 2.14 illustrates how the turbine output decreases as the percent ash in the coal increases.

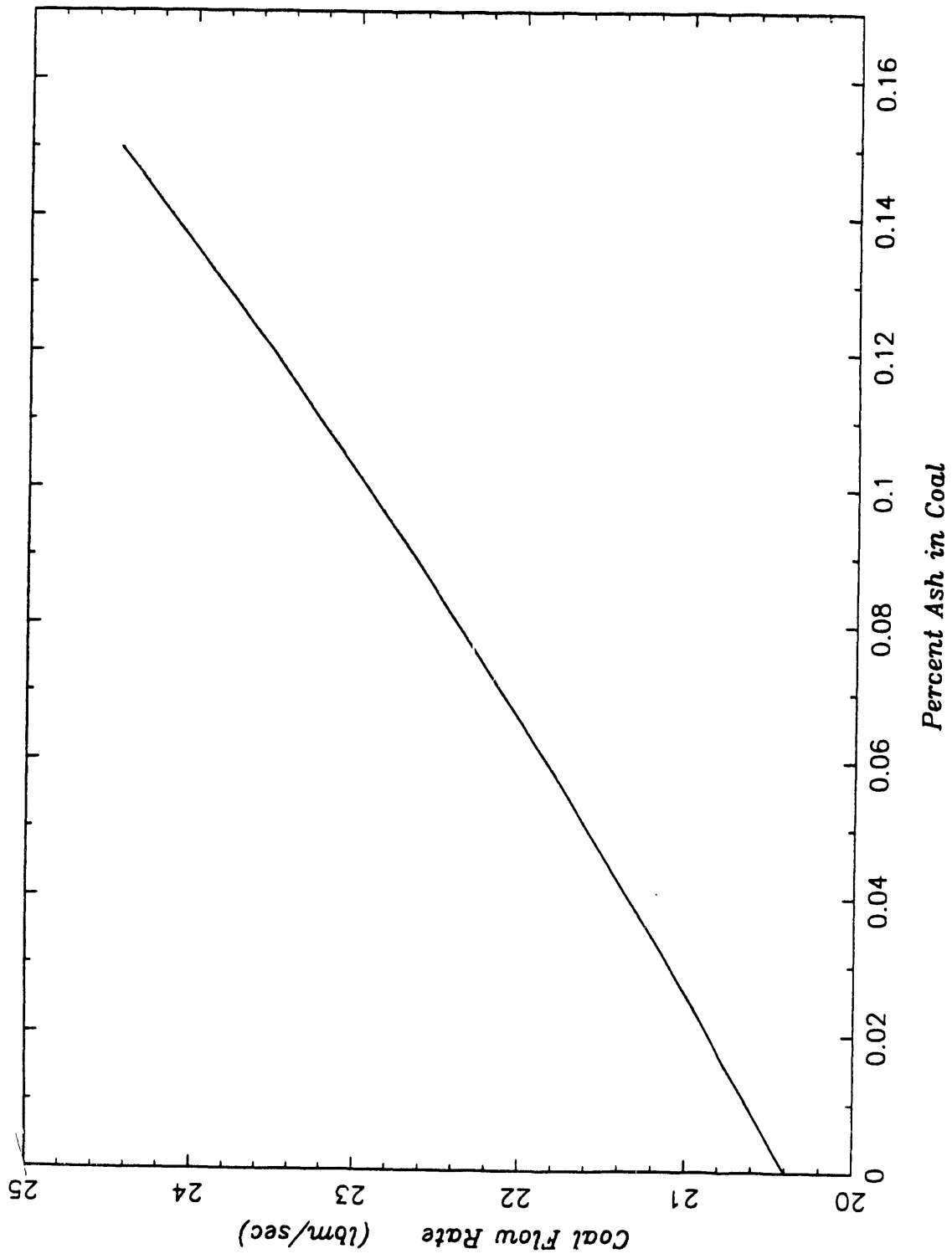


Figure 2.12 Coal Flow Rate versus Percent Ash in Coal

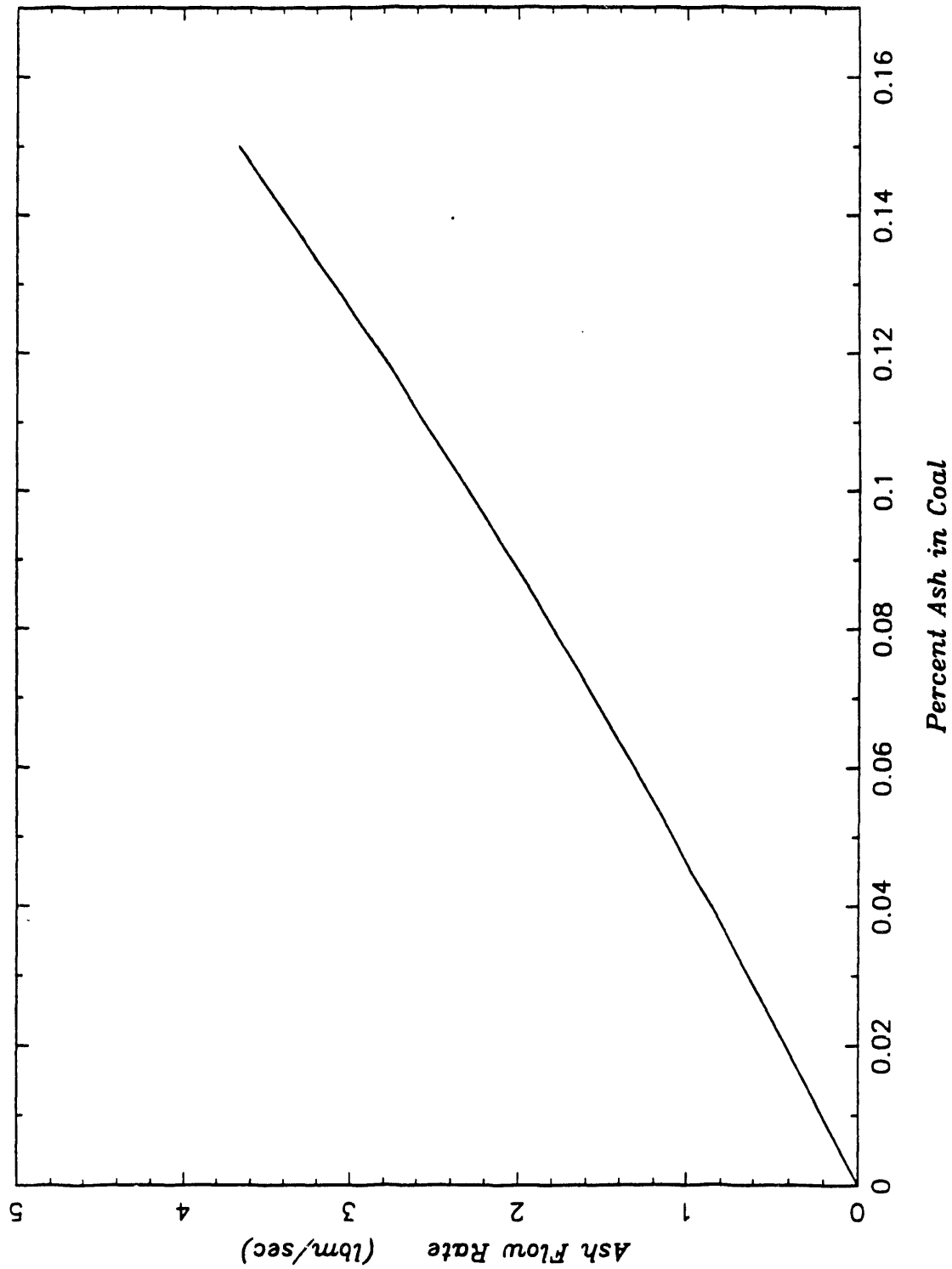


Figure 2.13 Ash Flow Rate versus Percent Ash in Coal

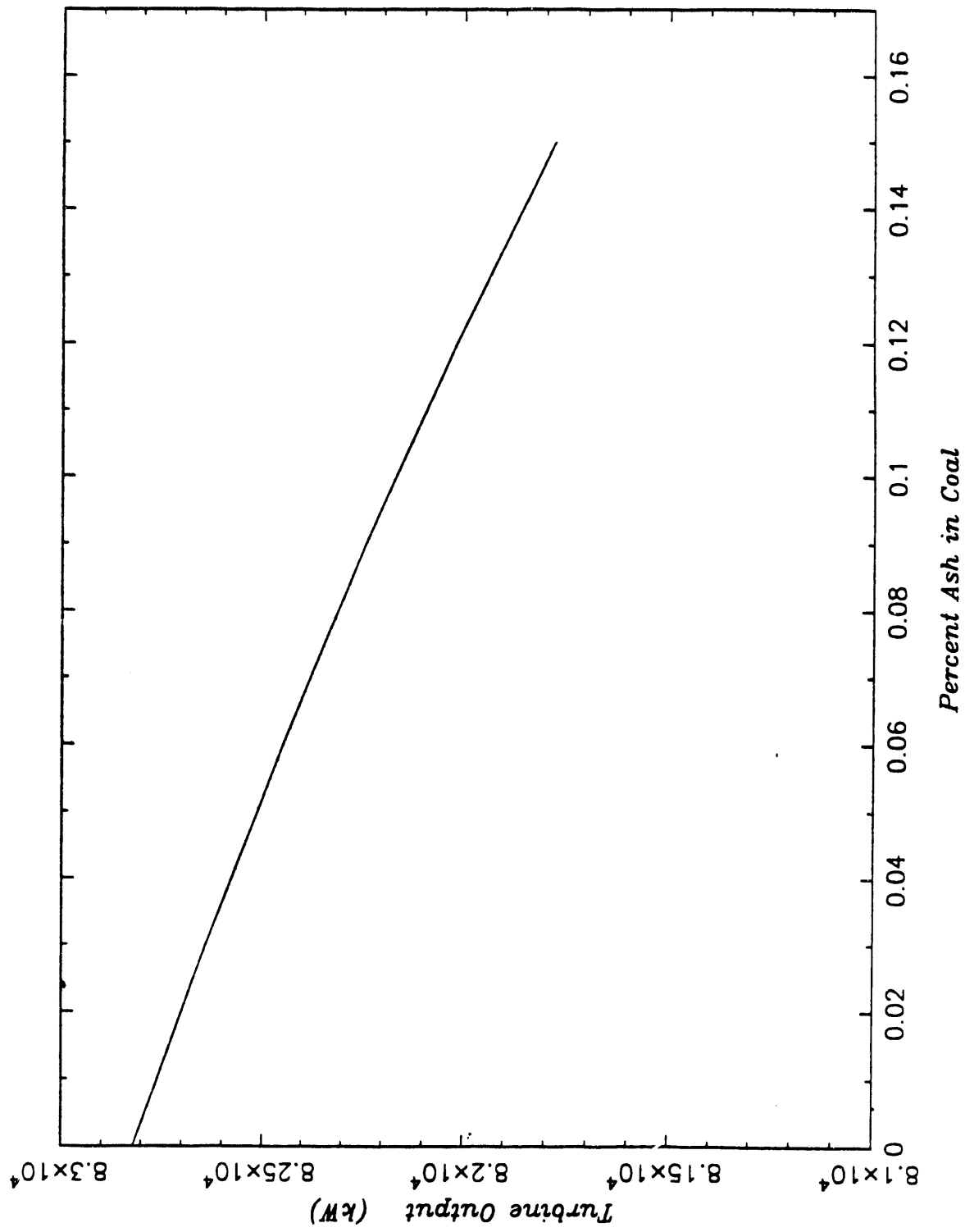


Figure 2.14 Turbine Output versus Percent Ash in Coal

### 3. FIXED BED COAL GASIFIER COMPUTER MODEL

A computer code, known as the Wen II model developed by Wen, Chen, and Onozaki [3] has been modified and applied to predict the performance of fixed bed gasifiers for the Standardized Air Blown Coal Gasifier/Gas Turbine Concept study. A copy of the source code was provided by the Morgantown Energy Technology Center (METC). Engineers at METC cautioned that the code employs several approximations that affect accuracy. A METC report [4] documents work done to evaluate the limitations of the model, and improved computer models of fixed bed gasifiers are now being developed under METC sponsorship. However, METC believes that the Wen II model is probably the best that is currently available.

Following a briefing at METC on limitations of the Wen II model, we studied the source code in detail, and ran several test cases to determine how model predictions compare to measurements. A decision was made to modify the Wen II code as described below before we made additional computer runs to predict gasifier performance.

- 1) The program would be modified to correct unrealistic calculations of quantities of tar and gas produced in the devolatilization of coal. The Wen II model contains equations for pulverized coal with particle diameters less than 0.01 centimeters. It predicts unrealistic results (such as negative mass flux) when applied at conditions needed for the present work. A decision was made to remove approximations in the model made to simplify mathematics, and to include effects such as temperature gradients within the solid that become important when the particle diameter is greater than 0.01 centimeters. The program would also be modified to account for energy needed to dry moisture from the coal prior to devolatilization.



- 2) The Wen II program would be modified so that it computes separate temperatures for the solid and gaseous phases. The model assumes that the solid and gas are at the same temperature. This permits a considerable simplification in mathematics. However, several investigations have expressed concern that this assumption may lead to serious errors in calculations for fixed bed gasifiers. Although it is expected that the temperatures of the solid and gas will be nearly the same at most locations in the gasifier, small difference may be important because reaction rates are very sensitive to temperature. The temperature differences may be significant in fixed bed gasifiers because the coal particles are relatively large. The consequences of the assumption have not been previously investigated. Therefore a decision was made to program separate energy equations for the solid and gaseous phases into the Wen II model so that differences in the temperatures of the two phases could be taken into account.

The modifications to the Wen II program were made so as to minimize the effects on the program structure. This was done so that a person who is familiar with the program would be able to recognize immediately where changes have been made, and could easily check to see whether the changes have been programmed correctly. When programming new equations, we used the same nomenclature that is in the original program, and tried to introduce as few new terms as possible. When describing the program modifications below we shall begin by describing the way that calculations were made in the original program, and shall then describe how the program has been modified. We discovered a few errors in the original program which we shall also discuss below when describing the program modifications.

### 3.1 GAS AND SOLID PHASE TEMPERATURE CALCULATIONS

#### 3.1.1 Method Before Modifications

The Wen II program assumes that the gas and solid are at the same temperature. This common temperature is calculated by solving the following steady state energy equation:

$$\left[ (\dot{m}c_p)_{gas} + (\dot{m}c_p)_{solid} \right] \frac{dT}{dz} = A \sum_{i=1}^6 H_i R_i + h\pi D(T_w - T) \quad (3.1)$$

The equation states that the increase in sensible energy in the gas and solid comes from energy released in chemical reactions and heat gain from the walls of the gasifier.  $H_i$  is the heat of reaction "i" in (cal/mole),  $R_i$  is the rate of the reaction per unit volume (moles/sec/cm<sup>3</sup>),  $A$  is the cross sectional area of the gasifier (cm<sup>2</sup>), and  $D$  is the diameter of the gasifier (cm).

We believe, however, that there is an error in the way that Eqn. 3.1 was programmed in the Wen II model. The error involves the sign of  $(\dot{m}c_p)_{solid}$ . The sign should be positive as it is in Eqn. 3.1 when the flow of the solid and gas is parallel. In this case  $(dT/dz)$  would be zero when there are no chemical reactions and no heat transfer from the wall. However, the sign of  $(\dot{m}c_p)_{solid}$  should be negative when the solid and gas are in counterflow as they are in a fixed bed gasifier.

$$\left[ (\dot{m}c_p)_{gas} - (\dot{m}c_p)_{solid} \right] \frac{dT}{dz} = A \sum_{i=1}^6 H_i R_i + h\pi D(T_w - T) \quad (3.2)$$

With counterflow,  $(dT/dz)$  does not have to be zero when the right hand side of the equation vanishes. The gas and solid can change temperatures by transferring heat to each other. Also, the heat of reaction causes the gas temperatures to increase as the gas flows upwards, and the solid temperature to increase as the solid flow downwards. This sign error has probably not been detected earlier because  $(\dot{m}c_p)_{solid}$  is small in comparison to  $(\dot{m}c_p)_{gas}$ . However, the term is not negligible. The sign error would affect the gas exit temperature and other calculated performance parameters, such as carbon conversion, which depend on temperature.

### 3.1.2 METHOD AFTER MODIFICATIONS

This section will address the modifications to the program concerning gas and solid phase temperature calculations. In addition, a small change was made to incorporate the devolatilization time of the coal. Wen II assumes that the devolatilization time is negligible in comparison to the gasification time. This assumption is true for very small coal particles (.01 cm), but for larger particles (2 to 5 cm), the devolatilization time is not negligible.

We have deleted the single energy equation and now solve two energy equations to determine the individual temperatures of the solid and gas. The energy equations that are solved in the modified version of the Wen II program are:

$$(\dot{m}c_p)_{gas} \frac{dT_g}{dz} = f \cdot A \left( \sum R_i H_i + R_{coal} c_{p,s} (T_s - T_g) \right) + h\pi D(T_w - T_g) + h_c A \frac{6}{D_p} (1 - \epsilon)(T_s - T_g) \quad (3.3)$$

$$(\dot{m}c_p)_{solid} \frac{dT_s}{dz} = -(1 - f)A \left( \sum R_i H_i + R_{coal} c_{p,s} (T_s - T_g) \right) + h_c A \frac{6}{D_p} (1 - \epsilon)(T_s - T_g) \quad (3.4)$$

where  $f$  is the fraction of the energy released in chemical reactions that goes directly into the gas phase,  $(6 \div D_p)(1 - \epsilon)$  is the surface area of the solid phase per unit volume, and  $h_c$  is the heat transfer coefficient between the gas and solid. The heat transfer coefficient should account for the thermal resistance of the ash layer that separates the gas and the reacting solid.

The actual value of  $f$  is unknown. It is probable that  $f$  is nearly unity since most of the reactions occur at the surface of the solid or in the gas phase. However, the effects of the ash layer, and of reactions that occur within the pores of the solid will cause  $f$  to be less than one. Parametric analysis indicate that the computed temperatures are not strongly dependent on the value of  $f$ .

In the revised program, rates for all reactions that involved the char are computed at the average temperature of the solid and gas. Gas phase reactions rates are computed at the gas temperatures.

An iterative procedure is used to solve Eqn.'s 3.3 and 3.4. The iterative solution is necessary because the gas inlet temperatures is prescribed at the bottom of the gasifier while the solid temperatures is prescribed at the top. Equation 3.3 is solved first to obtain preliminary gas temperatures by integrating upwards though the gasifier and assuming that the solid temperature is the same as the gas temperature. Equation 3.4 is then solved to obtain preliminary solid temperatures by integrating downwards from the top of the

gasifier. Equation 3.3 is then solved a second time to obtain corrected gas temperatures. In the second solution of Eqn. 3.3, heat transfer between the gas and solid is computed based on the corrected gas temperature and the preliminary solid temperature. The solid temperature is then corrected by solving Eqn. 3.4. The procedure is repeated until the updated temperatures agree with the temperature from the previous iteration.

Our experience indicates that convergence always occurs within a few iterations. We initially tried an iteration procedure that was very unstable. The unstable solution technique was one in which we attempted to guess the solid outlet temperature so that Eqn.'s 3.3 and 3.4 could be solved simultaneously. We anticipated that we could correct the assumed value of the solid outlet temperature until the computed solid temperature at the top of the gasifier matched the known solid inlet temperature. However the solution proved to be extremely sensitive to the assumed value of the solid outlet temperature. Underflows and overflows of exponents occurred for very reasonable trial values.

The heat transfer coefficient is computed using a correlation from the literature [8] for flow in packed beds.

$$h_c = 2.06 \cdot \frac{k}{D_p} Re^{.425} Pr^{.333} \quad (3.5)$$

This correlation does not account for the thermal resistance of the ash layer. By using a shape factor for conduction through a spherical shell [9], the heat transfer coefficient can be modified to compensate for the ash layer. The conduction heat transfer coefficient for the ash layer is given by:

$$h_{ash} = \frac{2 \cdot k_{ash}}{D_p} \left( \frac{\frac{r_i}{r_o}}{1 - \frac{r_i}{r_o}} \right) \quad (3.6)$$

where:

- $k_{ash}$   $\equiv$  Conduction Heat transfer coefficient for the ash layer.
- $D_p$   $\equiv$  Diameter of the coal particle.
- $r_i$   $\equiv$  Radius of the coal particle.
- $r_o$   $\equiv$  Radius of the coal particle plus the ash layer

The overall heat transfer coefficient is given by:

$$h_{overall} = \frac{1}{\frac{1}{h_c} + \frac{1}{h_{ash}}} \quad (3.7)$$

Since the surface area of the solid is very large, the calculated values of the solid and gas temperatures are nearly equal throughout most of the gasifier. Near the top, the gas temperature is greater than the solid temperature. Near the bottom of the gasifier, the solid temperature is greater than the gas temperature.

### 3.1.3 Programming the Changes

The solution to the energy equation for the gas phase is obtained by employing a numerical non-linear differential equation solver as in the original Wen II model. However, a semi analytical method was used to solve the solid phase energy equation. The numerical differential equation solver would have required small step sizes for this equation because  $(\dot{m}c_p)_{solid}$  is small. The solids energy equation, Eqn. 3.4, is rewritten in the following form.

$$\frac{dT_s}{dx} + \frac{h_c A}{(\dot{m}c_p)_s} T_s = \frac{h_c A}{(\dot{m}c_p)_s} T_g + \frac{1-f}{(\dot{m}c_p)_s} A \sum_i H_{R,i} \quad (3.8)$$

The solution to this non homogeneous differential equation contains two parts, the complementary solution and the particular solution. The solution to the homogeneous equation is:

$$T_{s,comp} = C \exp\left(-\frac{h_c A}{(\dot{m}c_p)_s} x\right) \quad (3.9)$$

To obtain a particular solution, we assumed that the gas temperature could be represented by a linear function of  $x$  over small distances. The particular solution is:

$$T_{s,p} = A + \frac{T_{g,j} - T_{g,j-1}}{x_j - x_{j-1}} x \quad (3.10)$$

where  $A$  is a constant. Differentiating, we get:

$$\frac{dT_{s,p}}{dx} = \frac{T_{g,j} - T_{g,j-1}}{x_j - x_{j-1}} \quad (3.11)$$

Substituting Eqn.'s 3.10 and 3.11 into Eqn. 3.8, we get:

$$T_{p,s} = T_g + \frac{(1-f)\sum H_R}{h_c A} - \frac{(\dot{m}c_p)_s}{h_c A} \left[ \frac{T_{g,j} - T_{g,j-1}}{x_j - x_{j-1}} \right] \quad (3.12)$$

Adding the complementary and the particular solution, Eqn.'s 3.9 and 3.12 and solving for the integration constant, we get:

$$\frac{T_{s,j-1} - T_{g,j-1} + \left( \frac{T_{g,j-1} - T_{g,j}}{x_{j-1} - x_j} \right) \frac{(\dot{m}c_p)_s}{h_c A} - \frac{(1-f)\sum H_R}{h_c A}}{T_{s,j} - T_{g,j} + \left( \frac{T_{g,j-1} - T_{g,j}}{x_{j-1} - x_j} \right) \frac{(\dot{m}c_p)_s}{h_c A} - \frac{(1-f)\sum H_R}{h_c A}} = \exp \left[ -\frac{h_c A}{(\dot{m}c_p)_s} (x_{j-1} - x_j) \right] \quad (3.13)$$

This equation may be used to determine the solid temperature  $T_{s,j-1}$  given  $T_{s,j}$  and  $T_g$ . The solid temperature at the top of the gasifier, which is needed to initialize the solution of Eqn. 3.13, is obtained from an energy balance of the devolatilization zone. The heat transfer from the gas in the devolatilization zone is given by:

$$\begin{aligned} \dot{Q} = & D_{TAR} \cdot [h_{f,tarv} - h_{f,tarl} + c_{p,tar} (T_E - T_O)] \\ & + D_{GAS} \cdot [h_{f,gasv} - h_{f,tarl} + c_{p,gas} (T_E - T_O)] \\ & + D_{MOIST} \cdot [h_{fg} + c_{p,moist} (T_E - T_O)] \\ & + (COAL - D_{TAR} - D_{GAS} - D_{MOIST}) \cdot c_{p,solid} (T_C - T_O) \end{aligned} \quad (3.14)$$

also,

$$\dot{Q} = GAS \cdot c_{p,gas} (T_F - T_E) \quad (3.15)$$

where:

- $T_O$   $\equiv$  Temperature of the coal entering the gasifier
- $T_C$   $\equiv$  Temperature of the char leaving the devolatilization zone
- $T_F$   $\equiv$  Temperature of the coalgas entering the devolatilization zone
- $T_E$   $\equiv$  Temperature of the coalgas leaving the gasifier
- $h_{fg}$   $\equiv$  Heat of vaporization for water
- $GAS$   $\equiv$  The gas flowrate into devolatilization zone
- $D_{TAR}$   $\equiv$  The amount of tar generated during devolatilization
- $D_{GAS}$   $\equiv$  The amount of gas generated during devolatilization
- $D_{MOIST}$   $\equiv$  The amount of moisture generated during devolatilization

A third equation is needed since there are three unknowns,  $\dot{Q}$ ,  $T_E$ , and  $T_C$  in Eqn.'s 3.14 and 3.15. The additional equation describes the way that heat transfer depends on the difference between the solid and gas temperatures. Here it is convenient to employ the concept of heat exchanger effectiveness.

$$T_C = T_O + \epsilon(T_F - T_O) \quad (3.16)$$

In counterflow heat transfer the effectiveness,  $\epsilon$ , is given by

$$\epsilon = \frac{1 - \Gamma}{1 - C \cdot \Gamma} \quad (3.17)$$

where:

$$\Gamma = \exp((C - 1) \cdot N) \quad (3.18)$$

$$C = \frac{(\dot{m}c_p)_{solid}}{(\dot{m}c_p)_{gas}} \quad (3.19)$$

$$N = \frac{hA}{(\dot{m}c_p)_{solid}} \quad (3.20)$$

and where  $hA$  is the overall conductance in the devolatilization zone. Equations 3.14 through 3.16 were solved simultaneously to obtain a single expression for the gas exit temperature,  $T_E$ . After some algebraic manipulation the equation for the gas exit temperature,  $T_E$ , can be written in the following form.

$$T_E = \frac{(\dot{m}c_p)_{gas}T_F - A1 + C1 \cdot T_C}{B1 + (\dot{m}c_p)_{gas}} \quad (3.21)$$

where:

$$\begin{aligned} A1 = & D_{TAR} \cdot (h_{f,tar,v} - h_{f,tar,l} - c_{p,tar}T_O) \\ & + D_{GAS} \cdot (h_{f,gas} - h_{f,tar,l} - c_{p,gas}T_O) \\ & + D_{MOIST} \cdot (h_{f,g} - c_{p,steam}T_O) \\ & + (D_{TAR} + D_{GAS} + D_{MOIST} - COAL) c_{p,solid}T_O \end{aligned} \quad (3.22)$$

$$B1 = D_{TAR} \cdot c_{p,tar} + D_{GAS} \cdot c_{p,gas} + D_{MOIST} \cdot c_{p,steam} \quad (3.23)$$

$$C1 = (COAL - D_{TAR} - D_{GAS} - D_{MOIST}) c_{p,solid} \quad (3.24)$$

and  $COAL$  is the coal feed rate into the top of the gasifier.

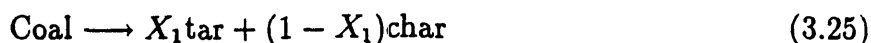
The overall conductance ( $hA$ ) depends on the height of the devolatilization zone. Also, the height of the devolatilization zone must be subtracted from the gasifier height to obtain the effective column height for conversion of char to gas. The height of the devolatilization zone depends on the devolatilization time and the coal feed rate. An iterative procedure is employed in the revised program to calculate the devolatilization time. Initially the devolatilization time and the height of the gasifier column required for devolatilization are neglected. Preliminary gas temperatures and compositions at the top of the column are then computed assuming that the entire column height is available for conversion of char to gas. These preliminary gas temperatures and compositions are then used to estimate the devolatilization time. The effective height of the gasifier column is then recalculated accounting for the column height needed for devolatilization, and all of the calculations are repeated.

### 3.2 DEVOLATILIZATION CALCULATIONS

The subroutines of the Wen II program that compute quantities of tar and gas formed during the devolatilization of coal were rewritten to remove approximations in mathematics and to account for effects such as temperature gradients within the solid which become important as the particle diameter increases. These changes were necessary because the original model predicted unrealistic results for conditions needed in the present work.

Devolatilization has been studied in great depth in recent years, and new theories and models have been developed. References [5] and [6] discuss recent studies. The Wen II program should eventually be revised to incorporate the new knowledge. However, new devolatilization theory has not been added to the model in the present work. We have only refined the mathematics in Wen's original model.

In the Wen II model coal devolatilizes to form tar and char:



where  $X_1$  is the mass fraction of the coal that forms tar. The model accounts for subsequent cracking of the tar, and also accounts for the fact that some of the tar reverts back to char in a



process called deposition. Thus the amount of tar produced during the devolatilization of coal is less than  $X_1$  and the amount of char remaining after devolatilization is greater than  $(1 - X_1)$ . The net amount of tar produced depends on differences in the rates of devolatilization, cracking, and deposition. Thus it changes with temperature, particle diameter, and other properties of coal. However, in the Wen II model, the net amount of char remaining after devolatilization is not a variable. It is a constant that is specified by the user, depending on the type of coal being fed into the gasifier. For example, when the coal is Illinois #6, the user should specify that 91.6% of the volatile matter is released during devolatilization. The net amount of char remaining after devolatilization is then determined from the proximate analysis of the coal.

$$\text{char} = 100 - 0.916 \times \text{VM} - \text{MOISTURE} - \text{ASH} \quad (3.26)$$

In this way the program permits experimental data to be incorporated into the devolatilization model. However, the user is also required to provide the value of the parameter  $X_1$  in Eqn. 3.25. Default values of  $X_1$  are included in the model for various coal types. One problem with the model is that the user specified or default value of  $X_1$  may not be consistent with the user specified value of char remaining after devolatilization. The Wen II program corrects for this problem as follows:

- 1) The amount of char remaining after devolatilization is calculated as the sum of  $(1 - X_1)$  and the amount of char deposited within the pores of the coal particle during the devolatilization process. The quantity of deposited char depends on the rate of deposition and the time required for devolatilization.
- 2) If the calculated amount of char remaining after devolatilization is less than the user specified value, the program assumes that its devolatilization calculations were incorrect. It reduces the amounts of tar and gas formed so that more of the devolatilization mass appears in the form of char.
- 3) If the calculated amount of char remaining after devolatilization is greater than the user specified amount, the excess char is assumed to react with hydrogen gas to form methane.



This correction sometimes leads to unrealistic results. The amount of hydrogen needed for the reaction may exceed the amount that is available in the gas stream. In this case the program predicts negative mass fractions of hydrogen and excessive mass fractions of methane in the product gas stream.

In the revised program, the value of  $X_1$  is calculated internally. It is not specified by the user. The program adjusts  $X_1$  until the calculated amount of carbon remaining after devolatilization agrees with the user specified amount. In this case corrections to the devolatilization calculations are not required.

### 3.3 TAR CALCULATIONS

The Wen II program calculates the quantity of tar produced by integrating rate equations. The net amount of tar produced is calculated as the amount devolatilized from the coal minus the amount lost through cracking and deposition. Thus, since the rate equations for devolatilization, cracking, and deposition depend on temperature, the quantity of tar produced from a given coal changes with the operating conditions. The model predicts that less tar will be produced when the coal particle diameter increases because the devolatilization process takes longer, and more time is thus available for cracking and deposition of tar. However, mathematical approximations are made in the model which limit the accuracy of the predictions.

Some boundary conditions were not satisfied when solving differential equations to calculate the mass of tar and gas evolved during devolatilization. Because of this, the calculated amount of tar flowing from the surface of the coal particle is not the same as the net amount generated within the particle after accounting for cracking and deposition. The discrepancy is significant as shown in Figure 3.1. The program has now been modified so that boundary conditions are satisfied and consistent results are obtained. This required extensive revision of the devolatilization subroutines. These revisions are described in detail below.

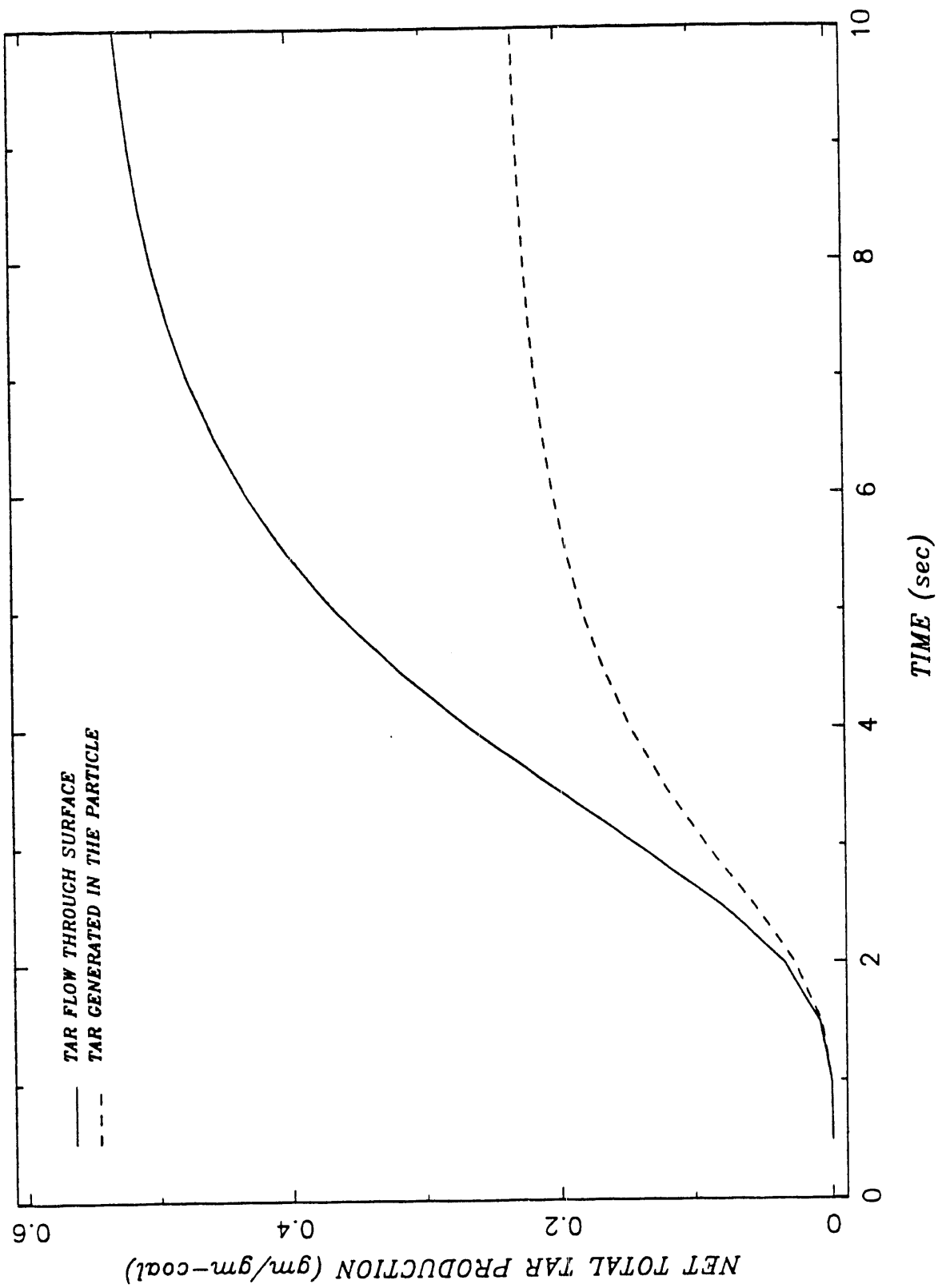


Figure 3.1: Comparison of Total Net Tar Generation for Flow Through the Particle Surface Versus Generation Inside the Particle.

We have also revised the subroutines to account for temperature gradients within the solid. Experiments indicate that devolatilization occurs in a relatively thin layer. The layer forms at the outside surface, which is heated first, and moves inwards as the temperature within the particle increases. The Wen II model, which was developed for pulverized coal assumes that the temperature within the particle is uniform. Thus, devolatilization occurs at the same time throughout the particle. No film develops. We have modified the program to account for temperature gradients within the particle. Figures 3.2 and 3.3 show some typical transients temperature profiles computed by the revised program. The energy equation solved to obtain these profiles accounts for heat needed to vaporize tars and gases. The program changes are documented below.

Initial results from the revised program indicated that very little tar was being formed in coal particles having diameters greater than 0.25 inches. The time required for devolatilization of these particles is sufficiently long that the program calculated that all of the tar cracked. We believe that the rate constant for cracking is too high. This constant was determined from experimental data for pulverized coal. Very high rates are required if cracking is to have any effect at all in this case. We have substituted rate constants from other sources [7], and the results are much more reasonable.

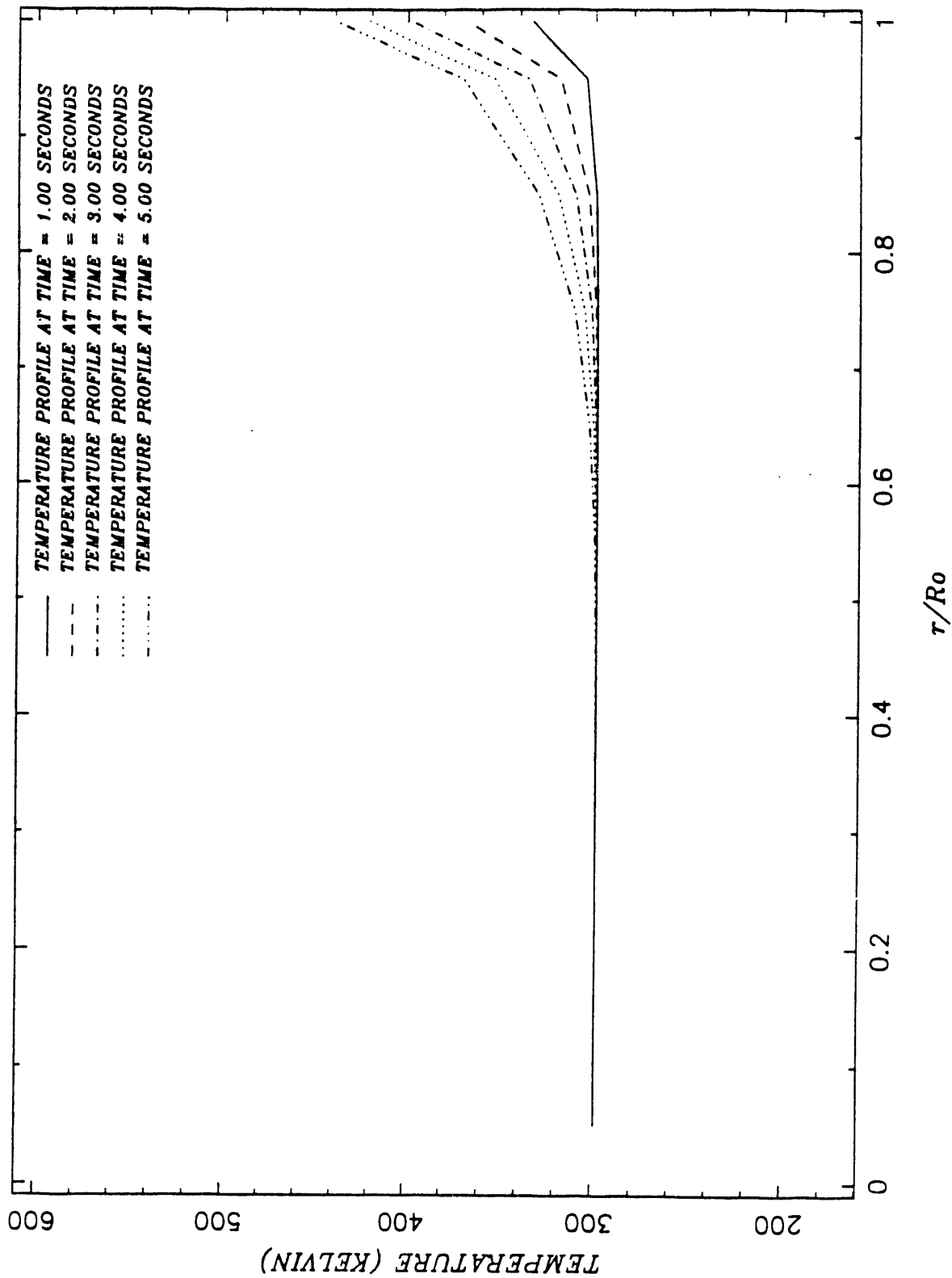
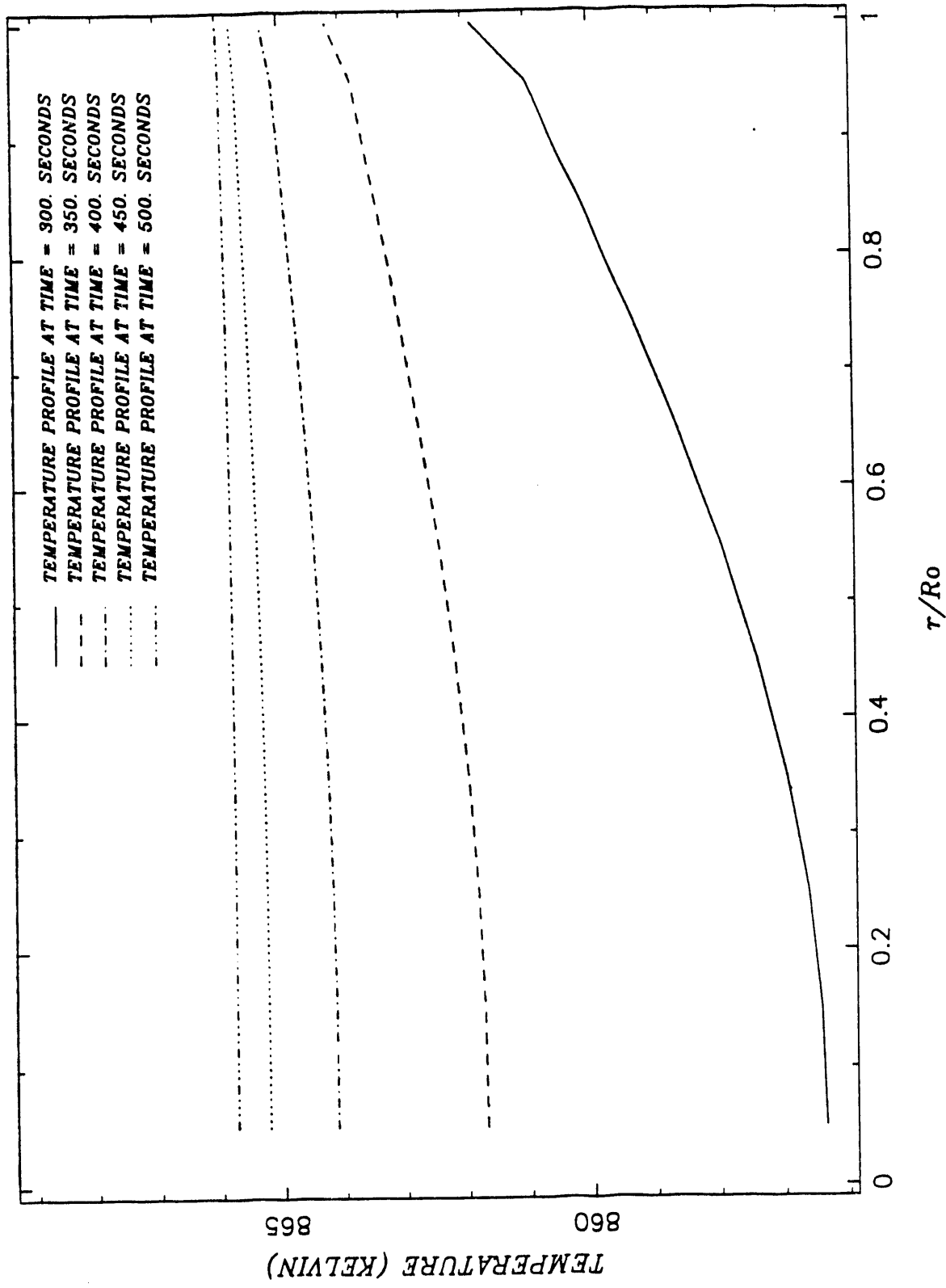


Figure 3.2: Comparison of Temperature Profiles at Various Times with a Particle Diameter of 0.5 inches.



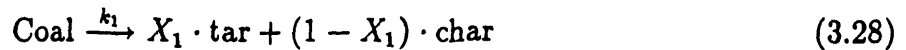
**Figure 3.3: Comparison of Temperature Profiles at Various Times with a Particle Diameter of 0.5 inches.**

### 3.3.1 METHOD BEFORE MODIFICATIONS

This section provides details of the way that the Wen II model accounted for the pyrolysis of coal. This will provide a starting point for discussing the program modifications. The Wen II model assumed that the three key reactions are: coal converted to tar and char, tar converted to product gas, and tar converted to char. These reactions were modeled using first order rate equations to give the reaction rates at various temperatures. Using conservation of species equations, both mass flux and species concentrations were calculated using a non-linear differential equation solver. These mass fluxes and concentrations were used in an overall energy equation and in coal particle weight loss calculation for each time interval.

The model bases the devolatilization of the coal particle on three reactions which occur simultaneously within the coal particle. The three reactions are devolatilization, cracking, and deposition.

\*Devolatilization:



$$\text{Rate} = k_{10} \cdot \exp\left(\frac{-E_{10}}{R_g T}\right) \cdot C_{\text{coal}} \quad (3.29)$$

\*Cracking:



$$\text{Rate} = k_{20} \cdot \exp\left(\frac{-E_{20}}{R_g T}\right) \cdot C_{\text{tar}} \quad (3.31)$$

\*Deposition:



$$\text{Rate} = k_{30} \cdot \exp\left(\frac{-E_{30}}{R_g T}\right) \cdot C_{\text{tar}} \quad (3.33)$$

The products of pyrolysis are tar, char, and product gas. Char is the undistillable material which remains in solid form; tar is material with a molecular weight greater than  $C_6$ , and appears in the vapor form; gas is material with a molecular weight lighter than  $C_6$ , i.e.  $CO$ ,  $CH_4$ ,  $C_2H_2$ ,  $H_2O$ , etc., and appears in the vapor form. The rates of formation for tar, product gas, and the inert gas can be expressed in the following equations:

$$R_{tar} = X_1 \cdot k_1 \cdot C_{coal} - (k_2 + k_3) \cdot C_{tar} \quad (3.34)$$

$$R_{gas} = k_2 \cdot C_{tar} \quad (3.35)$$

$$R_{inert\ gas} = 0 \quad (3.36)$$

The coal particle is considered to be a porous sphere which retains its integrity during pyrolysis. The conservation equation that is solved for the gaseous species, is:

$$\frac{1}{r^2} \cdot \frac{\partial}{\partial r} (r^2 \cdot N_i) = R_i \quad (3.37)$$

where  $R_i$  is the rate of generation of the species  $i$  due to chemical reactions.

$N_i$  is the molecular flux of the species and can be expressed as the sum of the rate of diffusion in the radial direction plus bulk flow through the pores.

$$N_i = -D_{eff,i} \cdot \frac{\partial C_i}{\partial r} + W_i \cdot \sum_j N_j \quad (3.38)$$

$W_i$ , which is the weight fraction of species  $i$ , can be expressed in the following equation:

$$W_i = \frac{C_i}{\sum_j C_j} \quad (3.39)$$

The following equation is solved to determine the solid concentration.

$$\frac{dC_1}{dt} = R_1 \quad (3.40)$$



Finally, the flow of the gaseous species  $i$  across the gas film is given by:

$$N_i = k_{g,i} \cdot [C_{i,s} - C_{i,b}] \quad (3.41)$$

where:

$C_{i,s}$   $\equiv$  concentration of the gas species  $i$  at the surface.

$C_{i,b}$   $\equiv$  concentration of the gas species  $i$  at the bulk gas stream.

$k_{g,i}$   $\equiv$  is the mass-transfer coefficient across the gas film which can be estimated from an appropriate mass-transfer correlation.

Equations 3.37 and 3.38 are coupled first order differential equations. One of the required boundary conditions is provided by Eqn. 3.41. The second boundary condition is  $N_i = 0$  at  $r = 0$ . In the Wen II program, the second boundary condition is not enforced. The coupled differential equations are solved by integrating numerically from the outside of the particle inwardly to  $r = 0$ . Thus, both boundary conditions are needed at the outside radius. The Wen II model estimates the unknown boundary condition for  $N_i$  at the surface, so that the integration can be performed, but does not correct the trial value so that  $N_i = 0$  at  $r = 0$ . As a result, the program predicts inconsistent and erroneous values for the mass flux. The net amount of tar produced inside the particle accounting for cracking and deposition is not the same as the amount of tar flowing through the surface of the particle. Figure 3.4 which graphs the tar mass flux predicted by the Wen II model, shows that the mass flux does not approach zero at the center of the particle.

### 3.3.2 METHOD AFTER MODIFICATIONS

The first item that was changed was the conservation equation for the gaseous species,  $i$ . The previous conservation equation neglected the amount of mass stored inside the particle. By neglecting the storage term, the conservation equation became an ordinary differential equation (ODE) which could be solved with a standard ODE solver. Including the storage

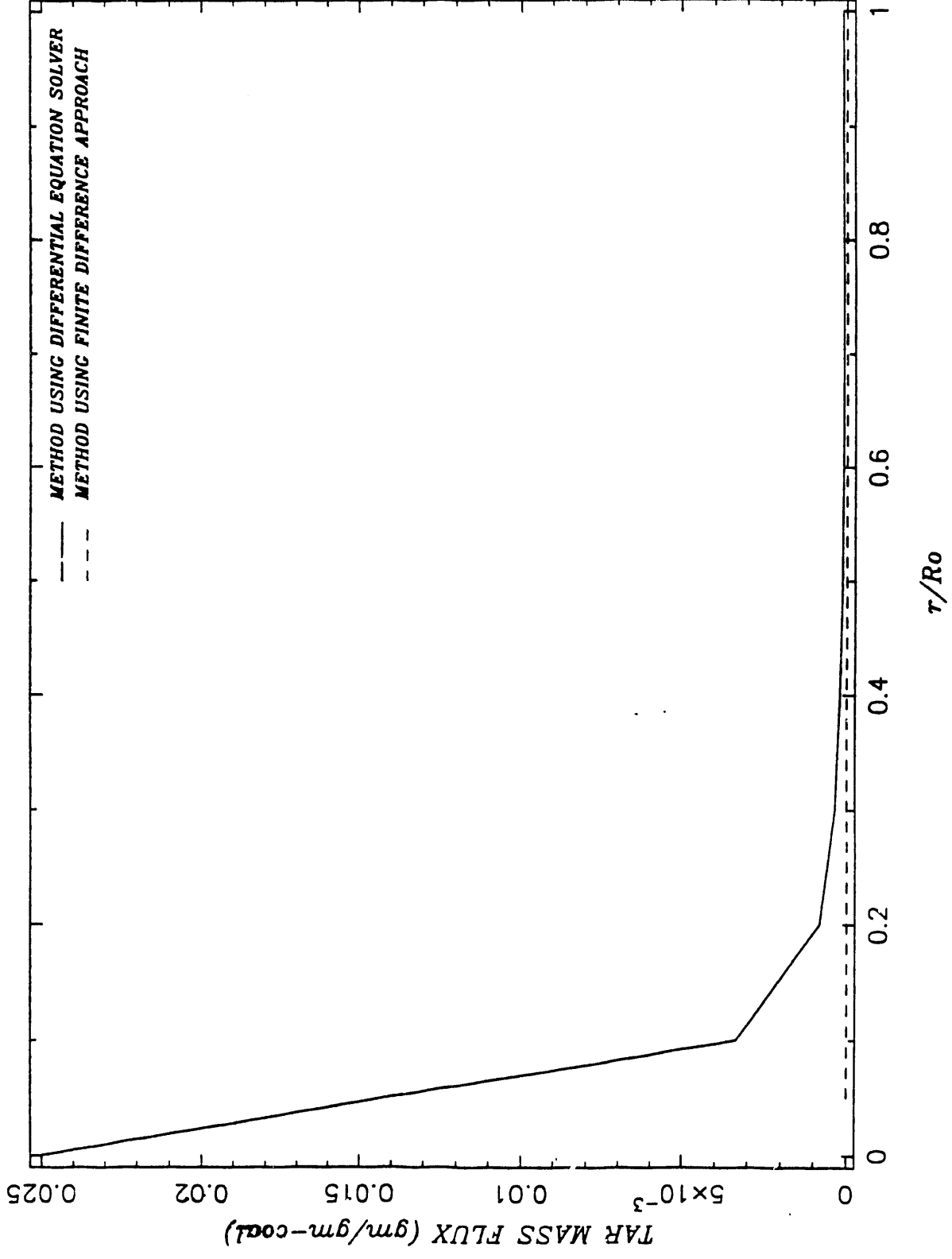


Figure 3.4: Comparison of Tar Mass Flux.

term in the gaseous species conservation equation results in the following partial differential equation:

$$\frac{1}{r^2} \cdot \frac{\partial}{\partial r} (r^2 \cdot N_i) + \frac{\partial C_i}{\partial t} = R_i \quad (3.42)$$

A finite difference approach was employed to solve the partial differential equation. After the coal particle was divided into shells, appropriate equations for species conservation and mass flux were derived for each shell. The concentration of species,  $i$ , is calculated at the midpoint of the shell and the mass flux of the species is calculated at the shell boundary.

Equations are programmed for three types of nodes. The node equations are for the center shell, the middle shells, and the outer shell. Starting with a mass balance, the rate of storage equations were derived; and then using Fick's law, the subsequent mass-flux equations were derived. The following equations were developed in this manner.

For Node # 1:

$$N'_1 \cdot 4\pi r_1^2 = \left( R_1 - \frac{dC_i}{dt} \right) \cdot \frac{4}{3} \pi r_1^3 \quad (3.43)$$

$$\frac{C'_1 - C_i}{\Delta t} = R_1 - \frac{3N'_1}{r_1} \quad (3.44)$$

$$N'_1 = -D_{eff} \left( \frac{C'_2 - C'_1}{\Delta r} \right) + (C'_1 + C'_2) \cdot \frac{\sum N_1}{\sum C_1 + \sum C_2} \quad (3.45)$$

For Node #  $i$  for  $1 < i < N$ :

$$(N'_i \cdot 4\pi r_i^2 - N'_{i-1} \cdot 4\pi r_{i-1}^2) = \left[ R'_i - \frac{dC_i}{dt} \right] \left[ \frac{4}{3} \pi \right] [r_i^3 - r_{i-1}^3] \quad (3.46)$$

$$\frac{C'_i - C_i}{\Delta t} = R'_i - 3 \left( \frac{N'_i r_i^2 - N'_{i-1} r_{i-1}^2}{r_i^3 - r_{i-1}^3} \right) \quad (3.47)$$

$$N'_i = -D_{eff} \left( \frac{C'_{i+1} - C'_i}{\Delta r} \right) + (C'_i + C'_{i+1}) \cdot \frac{\sum N_i}{\sum C_i + \sum C_{i+1}} \quad (3.48)$$

For Node #  $N$ :

$$(N'_N \cdot 4\pi r_N^2 - N'_{N-1} \cdot 4\pi r_{N-1}^2) = \left[ R'_N - \frac{dC_N}{dt} \right] \left[ \frac{4}{3}\pi \right] [r_N^3 - r_{N-1}^3] \quad (3.49)$$

$$\frac{C'_N - C_N}{\Delta t} = R'_N - 3 \left( \frac{N'_N r_N^2 - N'_{N-1} r_{N-1}^2}{r_N^3 - r_{N-1}^3} \right) \quad (3.50)$$

$$N'_N = kg \cdot [C'_N - C'_B] + C'_N \frac{\sum N_N}{\sum C_N} \quad (3.51)$$

These equations are the governing mass-flux and concentration equations used to determine the amount of tar and gas generated during devolatilization. They were solved implicitly using a tridiagonal matrix solver as described below. It should be noted that Eqn. 3.51, which prescribes the molecular flux at the boundary, is different from Eqn. 3.41, which is the corresponding equation in the Wen II model. The Wen II model neglects the bulk flow out of the particle surface.

The energy equation solved to determine the temperatures inside the particle accounts for energy transfer due to conduction, convection, and heats of reaction. A finite difference energy equation was generated for a spherical control volume. The general equation is:

$$\begin{aligned} c_p \cdot \left( \frac{T'_i - T_i}{\Delta t} \right) \left( \frac{4\pi (r_i^3 - r_{i-1}^3)}{3} \right) = \\ -k_{coal} \left( \frac{T'_i - T'_{i-1}}{\Delta r} \right) 4\pi r_{i-1}^2 + k_{coal} \left( \frac{T'_{i+1} - T'_i}{\Delta r} \right) 4\pi r_i^2 \\ + M_{j,i-1} \cdot \left( H_{f,j} + c_{p,j} \left( \frac{T'_{i-1} + T'_i}{2} \right) \right) 4\pi r_{i-1}^2 \\ - M_{j,i} \cdot \left( H_{f,j} + c_{p,j} \left( \frac{T'_i + T'_{i+1}}{2} \right) \right) 4\pi r_i^2 \\ + (R_{tar} + R_{gas}) \cdot (H_{f,tar,l} + c_{p,tar} T'_i) \cdot \left( \frac{4\pi (r_i^3 - r_{i-1}^3)}{3} \right) \end{aligned} \quad (3.52)$$

where:

- $i \equiv$  The shell of the particle number
- $j \equiv$  Species type; tar, gas, or inerts
- $M \equiv$  Species mass flux
- $R \equiv$  Species rate of generation
- $H_f \equiv$  Enthalpy of formation

The above equation applies to all of the shells except for the last one which is in contact with the ambient gas. The only change to the above equation for the final shell is the deletion of the positive (+) conduction term and the addition of the following convection term:

$$+ h \cdot (T_\infty - T'_i) 4\pi r_N^2 \quad (3.53)$$

Equation 3.52 with Eqn. 3.53 as a boundary condition determines the temperature profile across the particle. This equation was solved using the same tridiagonal matrix solver used for solving the species concentration equations.

The convective heat transfer coefficient in Eqn. 5.53 is computed from a correlation in the heat transfer literature [8] for flow in packed beds which is the same correlation that appears as Eqn. 3.5. The correlation requires average thermal properties for the gas mixture. A new subroutine (CONV) was written to calculate the heat transfer coefficient based on the average properties of the gas, the temperature, and the diameter of the particle.

### 3.1 PROGRAMMING THE CHANGES

This section converts the equations listed above into FORTRAN programming lines. It also discusses the changes to the affected subroutines called by the devolatilization subroutine. During the modification process, new subroutines were written to replace the devolatilization subroutine. The new subroutines are included in the appendices of this report.

The mass-flux and rate of storage equations were combined to get an overall conservation equation in terms of the old and new concentrations. Each shell had its own conservation



Since certain expressions appeared throughout the equations for a, b, c, and d, the following constants were defined for the ease of programming (assume  $r_o = 0$ ):

$$K1_i = \frac{3 \cdot \Delta t}{r_i^3 - r_{i-1}^3} \quad (3.58)$$

$$K2_i = r_i^2 \quad (3.59)$$

$$K3_i = \frac{Y1_i + Y2_i + Y3_i}{Y4_i + Y5_i + Y6_i + Y4_{i+1} + Y5_{i+1} + Y6_{i+1}} \quad (3.60)$$

$$K4_i = \frac{Y1_{i-1} + Y2_{i-1} + Y3_{i-1}}{Y4_i + Y5_i + Y6_i + Y4_{i-1} + Y5_{i-1} + Y6_{i-1}} \quad (3.61)$$

$$K5_i = \frac{(r_i^3 - r_{i-1}^3) \cdot \Delta t}{3} \quad (3.62)$$

$$K6_i = \left( \frac{Y1_i \cdot c_{p,tar}}{2} + \frac{Y2_i \cdot c_{p,gas}}{2} + \frac{Y3_i \cdot c_{p,inerts}}{2} \right) K2_i \Delta t \quad (3.63)$$

$$K7_i = (RATE_{tar} + RATE_{gas}) \cdot c_{p,tar} \cdot K5_i \quad (3.64)$$

$$K8_i = (Y1_i \cdot H_{f,tar,v} + Y2_i \cdot H_{f,gas} + Y3_i \cdot H_{f,inerts}) K2_i \Delta t \quad (3.65)$$

$$K9_i = (RATE_{tar} + RATE_{gas}) \cdot H_{f,tar,l} \cdot K5_i \quad (3.66)$$

$$K10_i = \frac{k_{coal} \cdot \Delta t \cdot K5_i}{\Delta r} \quad (3.67)$$

The following equations define the a's, b's, c's, and d's which were sent to TRIDI to find the new concentrations and temperatures. The equations will be grouped in 4's, the first set were applied to tar concentrations, the second were applied to gas concentrations, the third were applied to inerts concentrations, the fourth and final set were applied to the temperatures.

$$a_{tar} = -K1_i K2_{i-1} \cdot \left( \frac{D_{eff}}{\Delta r} + K4_i \right) \quad (3.68)$$

$$b_{tar} = 1 + K1_i K2_i \left( \frac{D_{eff}}{\Delta r} + K4_i \right) - K1_i K2_{i-1} \left( \frac{-D_{eff}}{\Delta r} + k4_i \right)$$

$$+(CK2_i + CK3_i) \cdot \Delta t \quad (3.69)$$

$$c_{tar} = K1_i K2_i \left( \frac{-D_{eff}}{\Delta r} + K3_i \right) \quad (3.70)$$

$$d_{tar} = X1 \cdot CK1_i \cdot CCOAL_i \Delta t + Y4_i \quad (3.71)$$

$$a_{gas} = -K1_i K2_{i-1} \cdot \left( \frac{D_{eff}}{\Delta r} + K4_i \right) \quad (3.72)$$

$$b_{gas} = 1 + K1_i K2_i \left( \frac{D_{eff}}{\Delta r} + K4_i \right) - K1_i K2_{i-1} \left( \frac{-D_{eff}}{\Delta r} + K4_i \right) \quad (3.73)$$

$$c_{gas} = K1_i K2_i \left( \frac{-D_{eff}}{\Delta r} + K3_i \right) \quad (3.74)$$

$$d_{gas} = CK2_i \cdot Y4_i \Delta t + Y5_i \quad (3.75)$$

$$a_{inerts} = -K1_i K2_{i-1} \cdot \left( \frac{D_{eff}}{\Delta r} + K4_i \right) \quad (3.76)$$

$$b_{inerts} = 1 + K1_i K2_i \left( \frac{D_{eff}}{\Delta r} + K4_i \right) - K1_i K2_{i-1} \left( \frac{-D_{eff}}{\Delta r} + K4_i \right) \quad (3.77)$$

$$c_{inerts} = K1_i K2_i \left( \frac{-D_{eff}}{\Delta r} + K3_i \right) \quad (3.78)$$

$$d_{inerts} = Y6_i \quad (3.79)$$

$$a_{temp} = -K6_{i-1} - K10_{i-1} \quad (3.80)$$

$$b_{temp} = \frac{c_{p,coal} \cdot K5_i}{\Delta t} + K6_i - K6_{i-1} - K7_i + K10_i + K10_{i-1} \quad (3.81)$$

$$c_{temp} = K6_i - K10_i \quad (3.82)$$

$$d_{temp} = \frac{c_{p,coal} \cdot K5_i \cdot Y7}{\Delta t} - K8_i + K8_{i-1} + K9_i \quad (3.83)$$

where:

$CK1, CK2 \equiv$  Rate constants for devolatilization, cracking

$Y1, Y2, Y3 \equiv$  Mass flux of tar, gas, inerts



$Y_4, Y_5, Y_6 \equiv$  Concentration of tar, gas, inerts  
 $Y_7 \equiv$  Temperature of particle

The above equations apply for nodes greater than one and less than N. The equations for nodes i and N, which are listed in the source code in Appendix A, were obtained by only a slight variation of these equations.

In order to calculate an appropriate heat transfer coefficient, values for Prandtl Number, conductivity, specific heat, and viscosity were needed. These values were determined for each individual gas,  $CO_2, H_2, \dots$ , and then averaged to obtain one value for each constant at the specified temperature. The Wen II program contained a subroutine to calculate the specific heats at a given temperature. In the new subroutine, CONV, Prandtl Numbers for each individual gas were averaged to obtain the Prandtl Number for the collective gas mixture. The conductivities for each gas were determined by using the following formula:

$$k_i = A + B \cdot T \quad (3.84)$$

The A's and B's were obtained by a curve fit to experimental data. The conductivities of individual gases were then averaged to obtain an effective conductivity of the gas mixture. The following equations were used to determine the heat transfer coefficient:

$$\mu_{avg} = \frac{Pr_{avg} \cdot k_{avg}}{c_{p,avg}} \quad (3.85)$$

$$Re = \frac{\sum_{i=1}^7 \dot{m}_i \cdot D_p}{A \cdot \mu} \quad (3.86)$$

$$Nu = \frac{2.06 Re^{.425} Pr^{.333}}{\text{bed voidage}} \quad (3.87)$$

$$h = \frac{Nu \cdot k_{avg}}{D_p} \quad (3.88)$$

where:

$D_p \equiv$  The diameter of the particle

$A \equiv$  The cross-sectional area of the gasifier

In test cases, we observed that the particle was still heating up when the tar production approached zero. Since the convergence criteria in the Wen II model was based on the tar

and gas generation during devolatilization, the subroutine would stop prematurely. We now base the convergence criteria on temperature only, such that the subroutine will stop when the temperature at the center of the coal particle is within 1 °C of the gas temperature entering the devolatilization zone.

### 3.4 Comparisons of Results

Below we discuss the results of several test cases that were run to assess the accuracy of the computer model predictions, and to determine how the gasifier performance depends on input parameters such as coal particle size and gasifier bed height. We also ran several test cases to determine the effects of injecting water into the inlet air stream. Water injection would cool the air and reduce the amount of steam needed in the gasification process. The first two cases were run to compare the results of the original Wen II code and the modified code. The last ten cases were run to determine how the gasifier performance depends on input parameters and to determine the effects of water injection into the air stream.

#### 3.4.1 Computer Model Predictions

In cases 1 and 2, experimental data from Ref. [7] are compared to the model predictions. The original model was run using the input data prescribed in Ref. [7], and the results were compared to results from the modified model and to the experimental data. Table 3.1 lists the input parameters for the two cases, with Table 3.2 listing the results.

As seen in Table 3.2, a discrepancy exists between calculated values of methane, hydrogen, tar production, and gas exit temperature. In the Wen II model most of the methane is produced during the devolatilization process, by hydrogasification according to Eqn. 3.27. The value of  $X_1$  in Eqn. 3.25 can be adjusted to make the model predictions agree with measurements. In the modified program the value of  $X_1$  is computed internally so that no hydrogasification occurs during the devolatilization process. The value of  $X_1$  could easily be set by the model so that any prescribed amount of methane is formed by hydrogasification.

Table 3.1 Input Data for Cases 1 and 2

Coal Feed Rate, lbs/hr	8058.00				
Proximate Analysis of Pittsburgh #8 Coal, Wt %					
Moisture	4.58	Volatile matter	37.37		
Ash	7.74	Fixed Carbon	50.31		
Ultimate Analysis of Pittsburgh #8 Coal, Wt %					
C	74.2	H <sub>2</sub>	5.0	O <sub>2</sub>	4.5
N <sub>2</sub>	1.4	S	2.5	H <sub>2</sub> O	4.58
Ash	7.74				
Heating Value of Coal, Btu/lb-wet coal	13441.80				
Steam Feed Rate, lbs/hr	26145.00				
Oxygen Feed Rate, lbs/hr	5005.00				
Oxygen Content, mole ratio	0.94				
Temperature of Coal, °F	77.00				
Temperature of Steam, °F	700.00				
Temperature of Oxygen, °F	700.00				
Temperature of Wall, °F	700.00				
Pressure, psig	350.00				
Bed Diameter, ft	10.00				
Bed Height, ft	10.00				
Initial Coal Particle Size, in	0.79				
Heat Transfer Coefficient, Btu/(ft <sup>2</sup> · hr · °R)	30.00				
Bed Voidage	0.40				

However, it is expected that most of the hydrogasification will occur following devolatilization. Other processes are probably responsible for methane formation during devolatilization. Alternative methods for methane formation in the devolatilization will need to be addressed in future model development.

The differences in the tar production are due to the tar cracking rate constants. The rate constant could be adjusted to make the model predictions agree with the experimental data. It should be noted that the Wen II model changes the coal particle diameter from 0.79 inch to 0.01 centimeter to obtain the results in Table 3.2. The Wen II model predicts zero tar production when the particle diameter is 0.79 inch. The modified code does not

arbitrary change the particle diameter.

The gas outlet temperature depends on the heat transfer coefficient at the water cooled walls of the gasifier. The heat transfer coefficient specified for Table 3.2 is  $h = 30 \text{ Btu}/(\text{hr ft}^2 \text{ }^\circ\text{F})$ . This value appears to be too high because the heat loss to the walls is greater than is typical of fixed bed gasifiers. In additional test cases described below for a Lurgi Mark IV gasifier, we found that a heat transfer coefficient of  $h_c = 17 \text{ Btu}/(\text{hr ft}^2 \text{ }^\circ\text{F})$  resulted in the correct heat loss to the water jackets. This value would raise the gas outlet temperature in Table 3.2. However we have not attempted to fine tune the model to match experimental results in Cases 1 and 2. We only wish to show the effects of the program modifications.

**Table 3.2 Comparison of Results to Experimental Data for Cases 1 and 2**

	Experimental Results	Original Code Results	Modified Code Results
Total Gas Output, lbs mole/hr	771.0	859.56	690.39
Product Gas Composition, mole %			
CO	16.8	15.54	16.83
CO <sub>2</sub>	31.3	30.01	30.34
H <sub>2</sub>	39.2	44.52	49.82
CH <sub>4</sub>	10.3	7.63	0.40
N <sub>2</sub>	1.6	1.57	1.95
H <sub>2</sub> S	0.7	0.73	0.91
Exit Steam Rate, lbs moles/hr	1189.0	1140.8	1250.9
Amount of Tar, lbs/hr	425.0	453.1	227.2
Temperature of Exit Gas, °F	1196.0	1208.5	1069.0
Maximum Temperature, °F	-	1942.0	1978.0
Carbon Conversion, %	98.9	98.5	99.1
Heating Value of Dry Gas, Btu/SCF	285.0	264.8	220.8
Heat Loss, 10 <sup>6</sup> Btu	-	74.2	70.8

Figures 3.5 through 3.11 are a direct comparison of temperature and methane, carbon monoxide, carbon dioxide, hydrogen, water/steam, and oxygen molecular fluxes respectively. The original and modified versions produced very similar results. The gasification maximum temperature was slightly higher and the gas exit temperature was slightly lower than that of the original version.

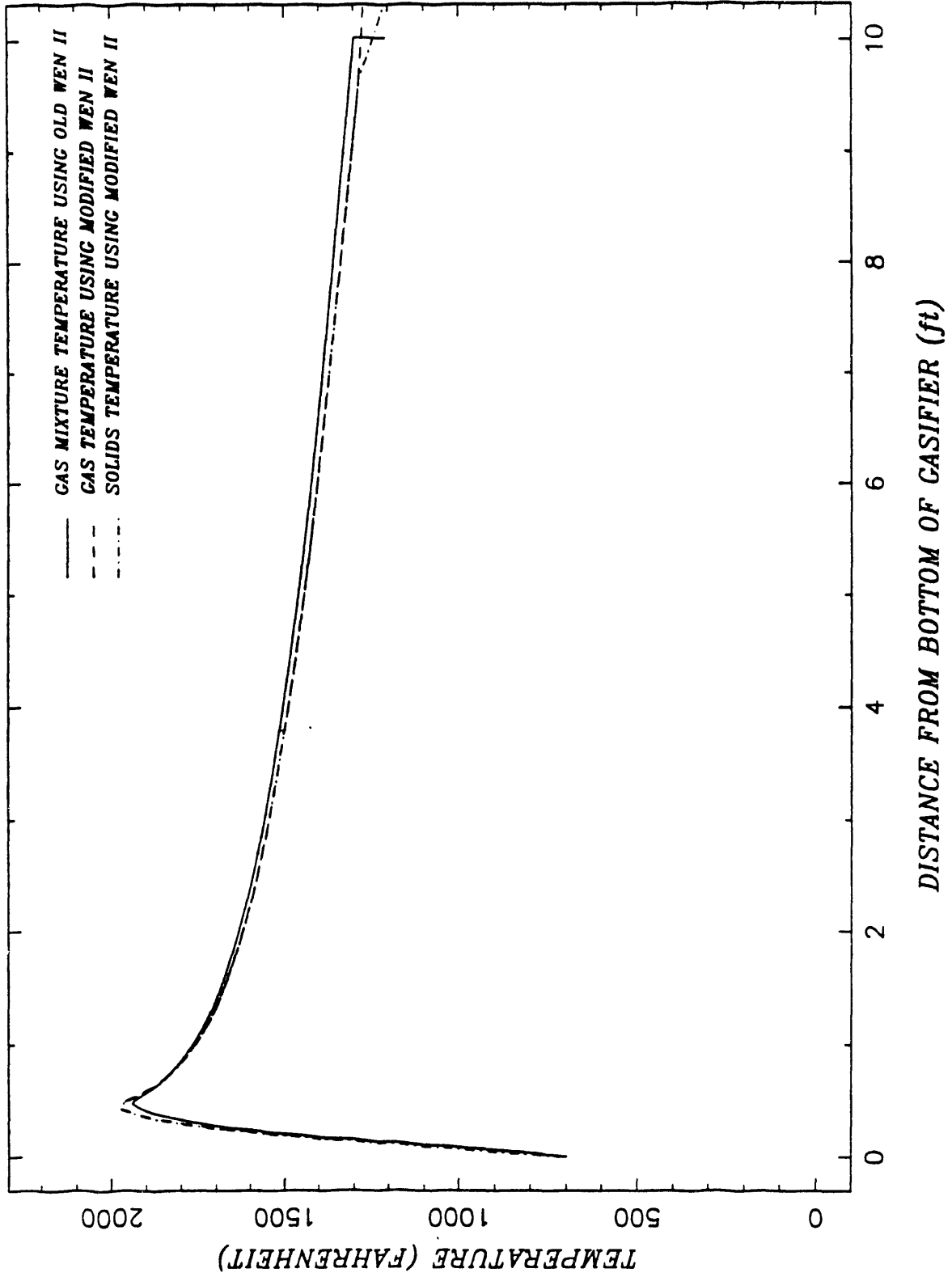


Figure 3.5: Comparison of Temperature Profiles.

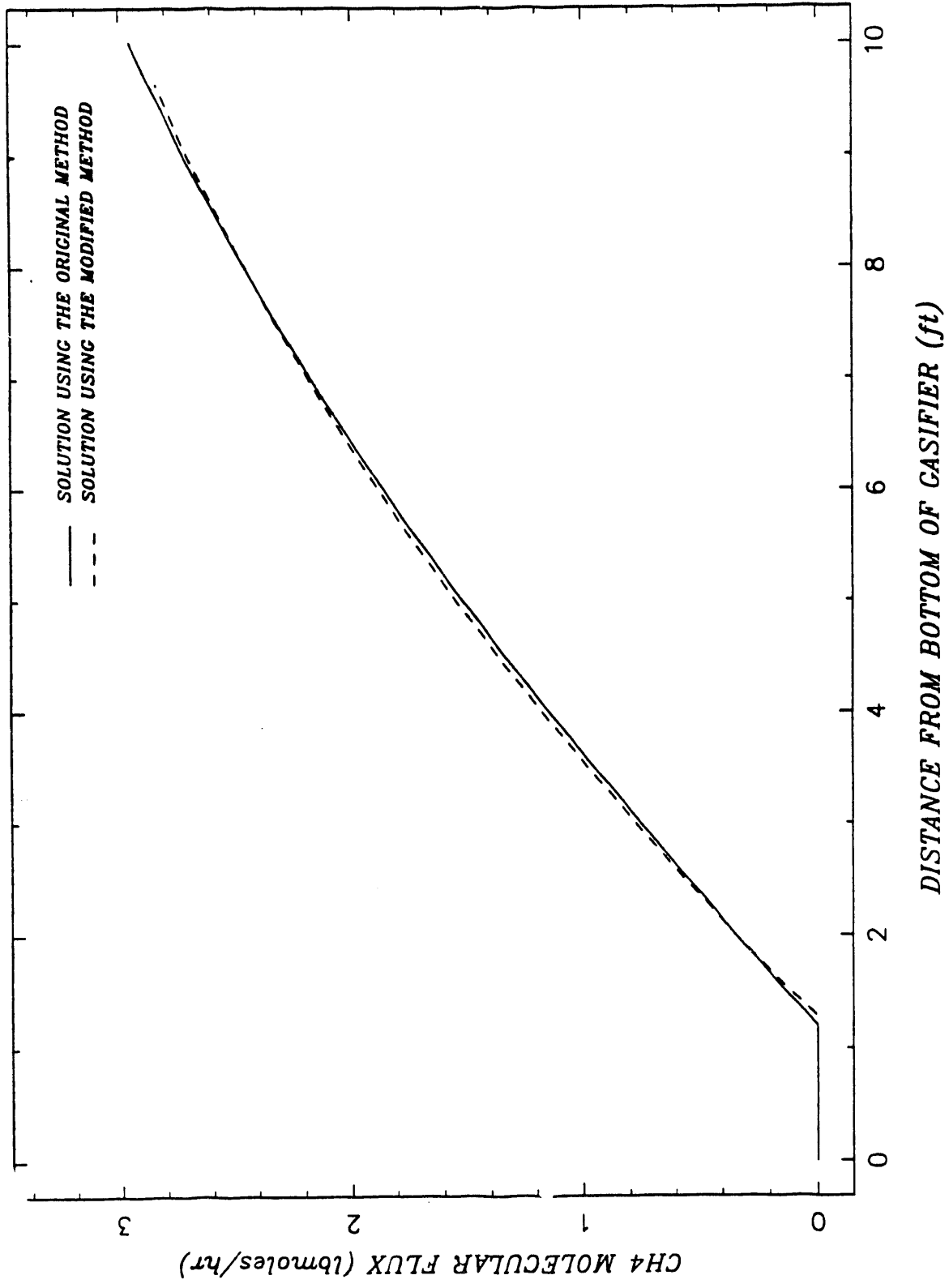
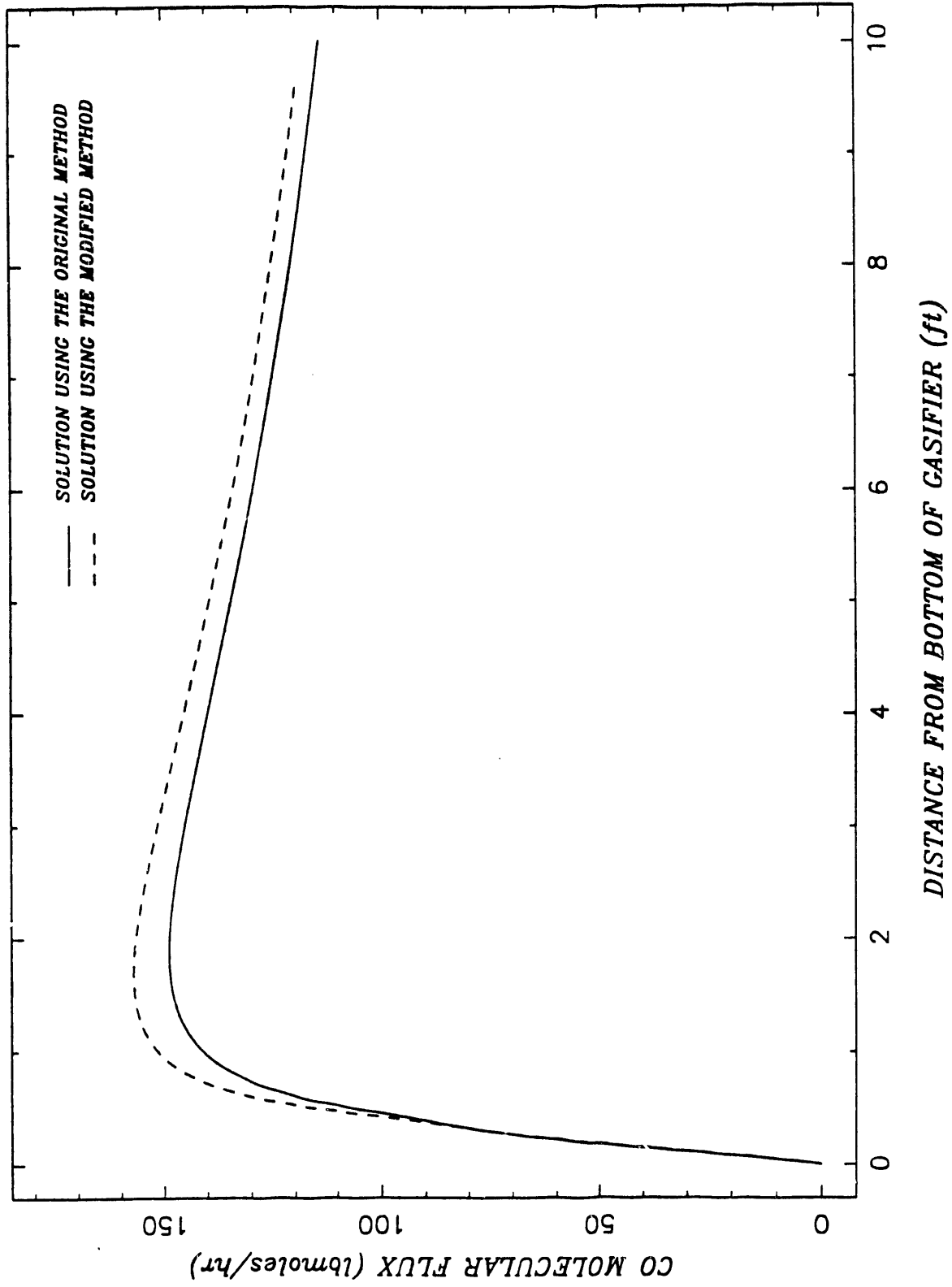


Figure 3.6: Comparison of Methane, CH<sub>4</sub>, Profiles.



3.7: Comparison of Carbon Monoxide, CO, Profiles.

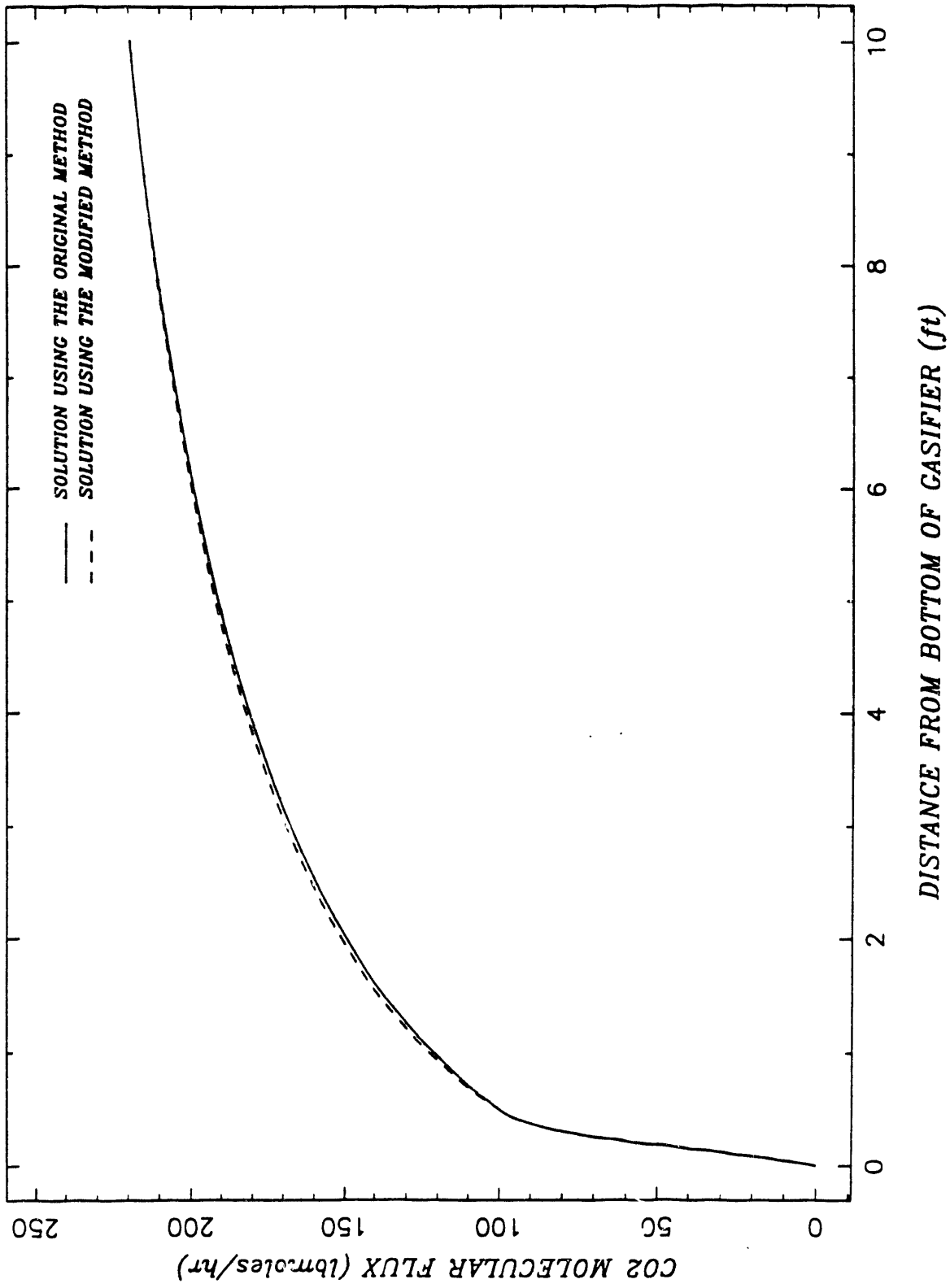


Figure 3.8: Comparison of Carbon Dioxide,  $CO_2$ , Profiles.



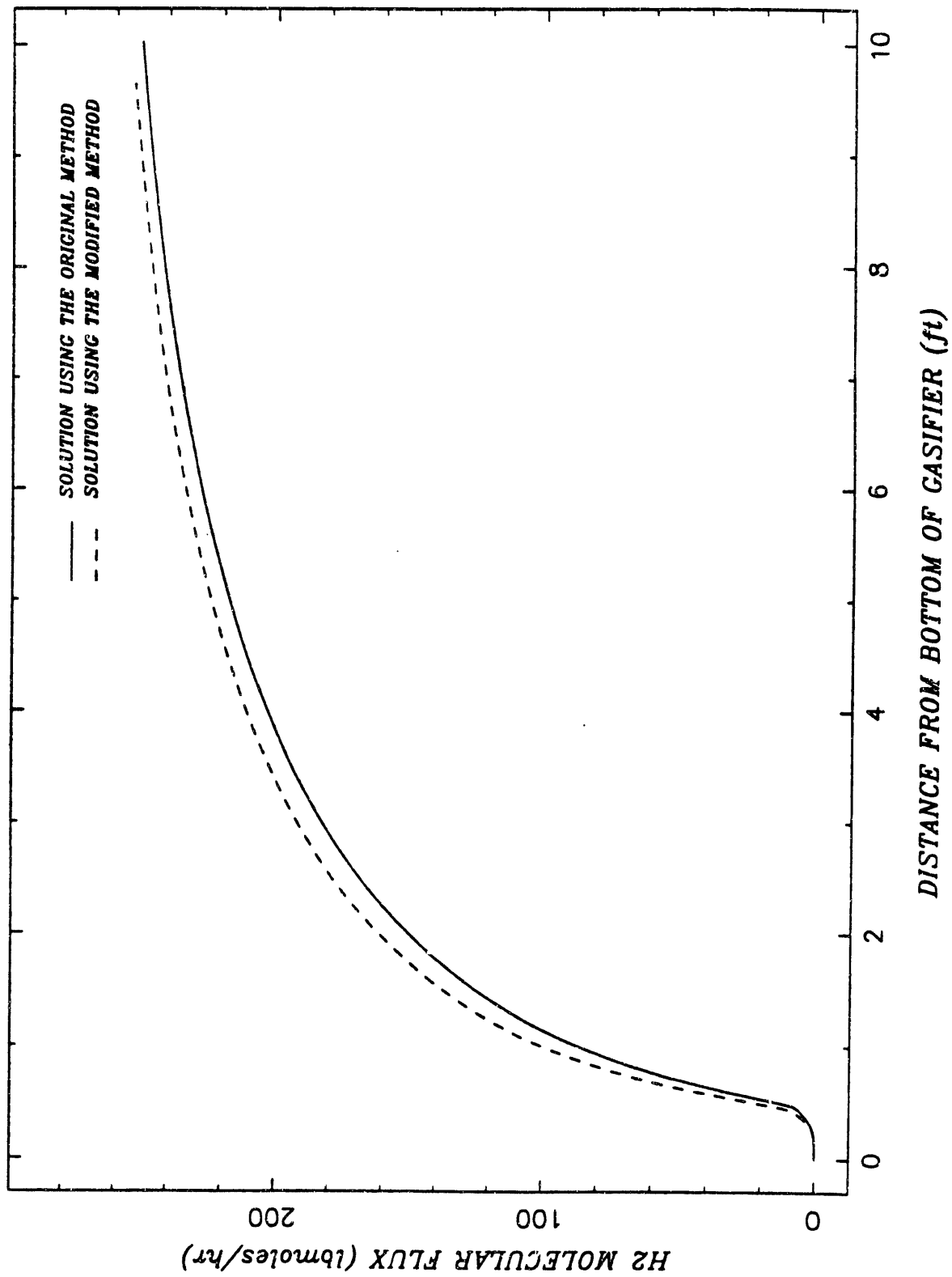


Figure 3.9: Comparison of Hydrogen, H<sub>2</sub>, Profiles.

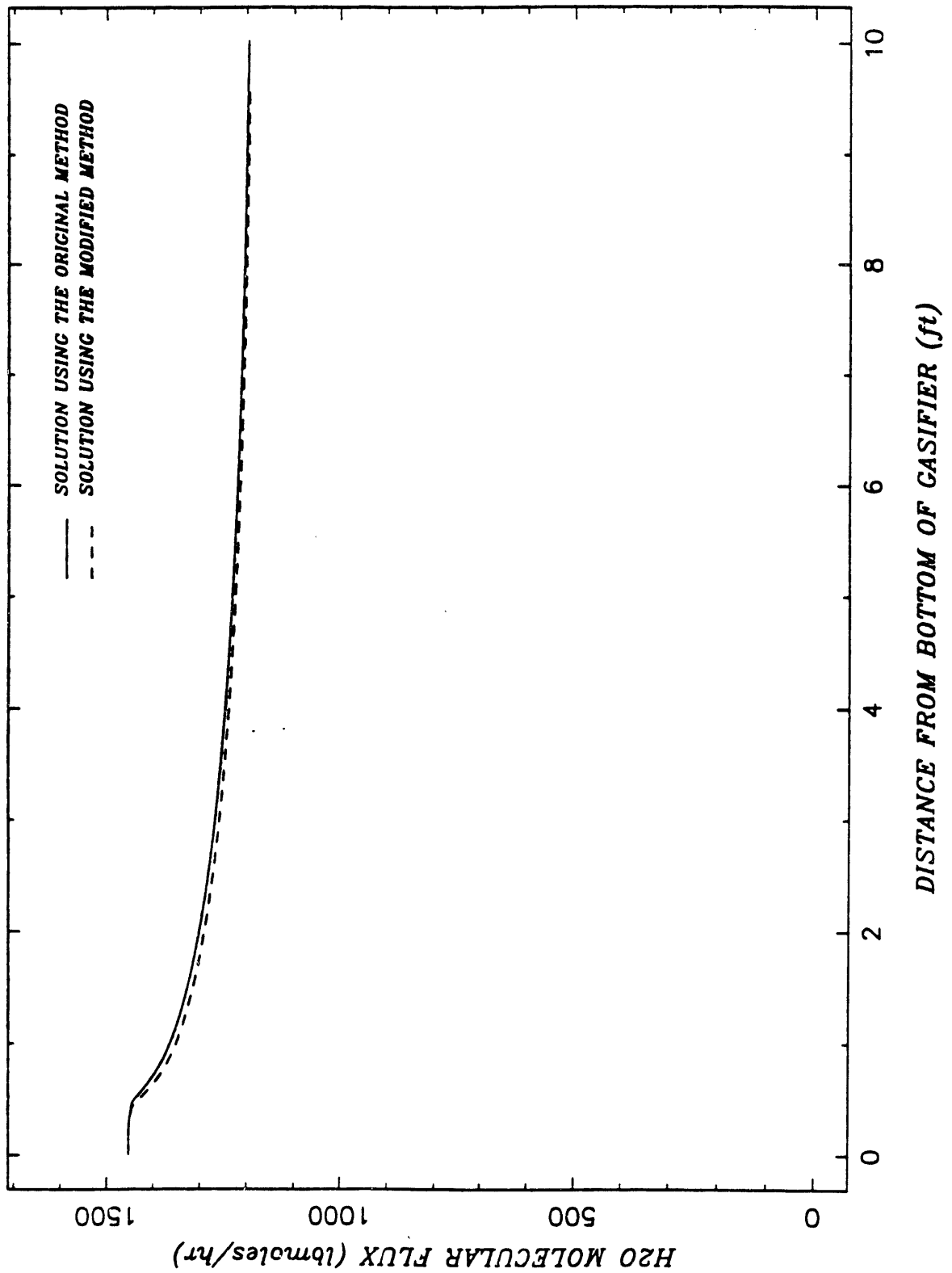


Figure 3.10: Comparison of Water/Steam, H<sub>2</sub>O, Profiles.

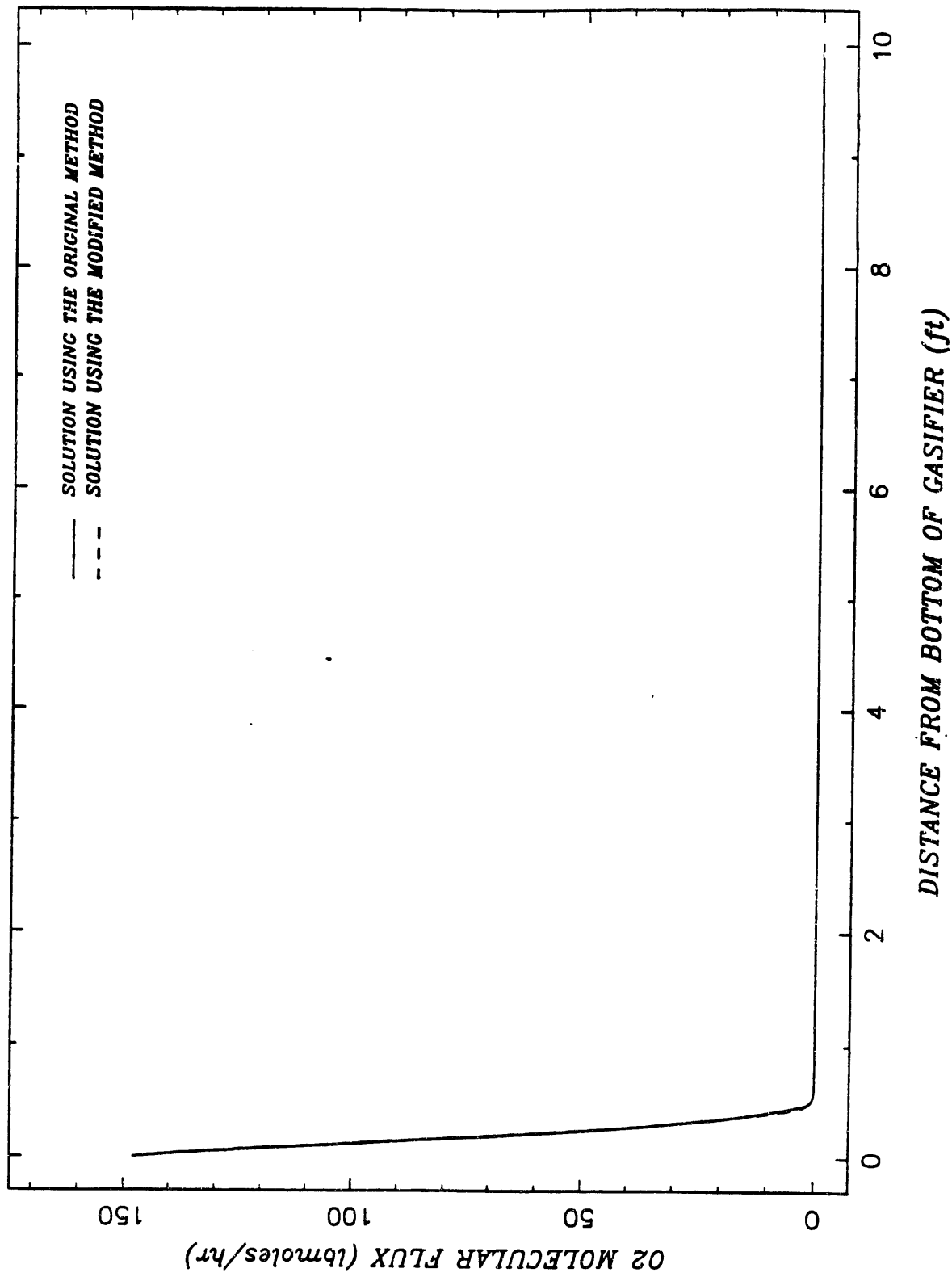


Figure 3.11: Comparison of Oxygen, O<sub>2</sub>, Profiles.

Figures 3.12 through 3.15 show the total net tar plots and the instantaneous net tar generation plots for both versions. These plots compare the tar flow through the surface to the tar generated inside the particle. The original version of Wen II has a large discrepancy between the tar generated in the particle and the flow through the surface. Figures 3.14 and 3.15, which are the plots for the modified version, show very good agreement between the tar generated in the particle and tar flow through the surface.

Figures 3.16 through 3.27 show the tar, gas, and inerts mass fluxes and concentrations at various times for both versions. As seen in these figures, the time for devolatilization in the modified version is much longer than that of the original version. This time difference can be attributed to the fact that the modified version used the actual coal diameter and accounted for coal particle heat up. As seen in Figure 3.25, the tar generation becomes zero in about a fourth of the time that it takes the particle to heat up to the gasification temperature. This time used in the heating of the particles affects the overall performance of the gasifier.

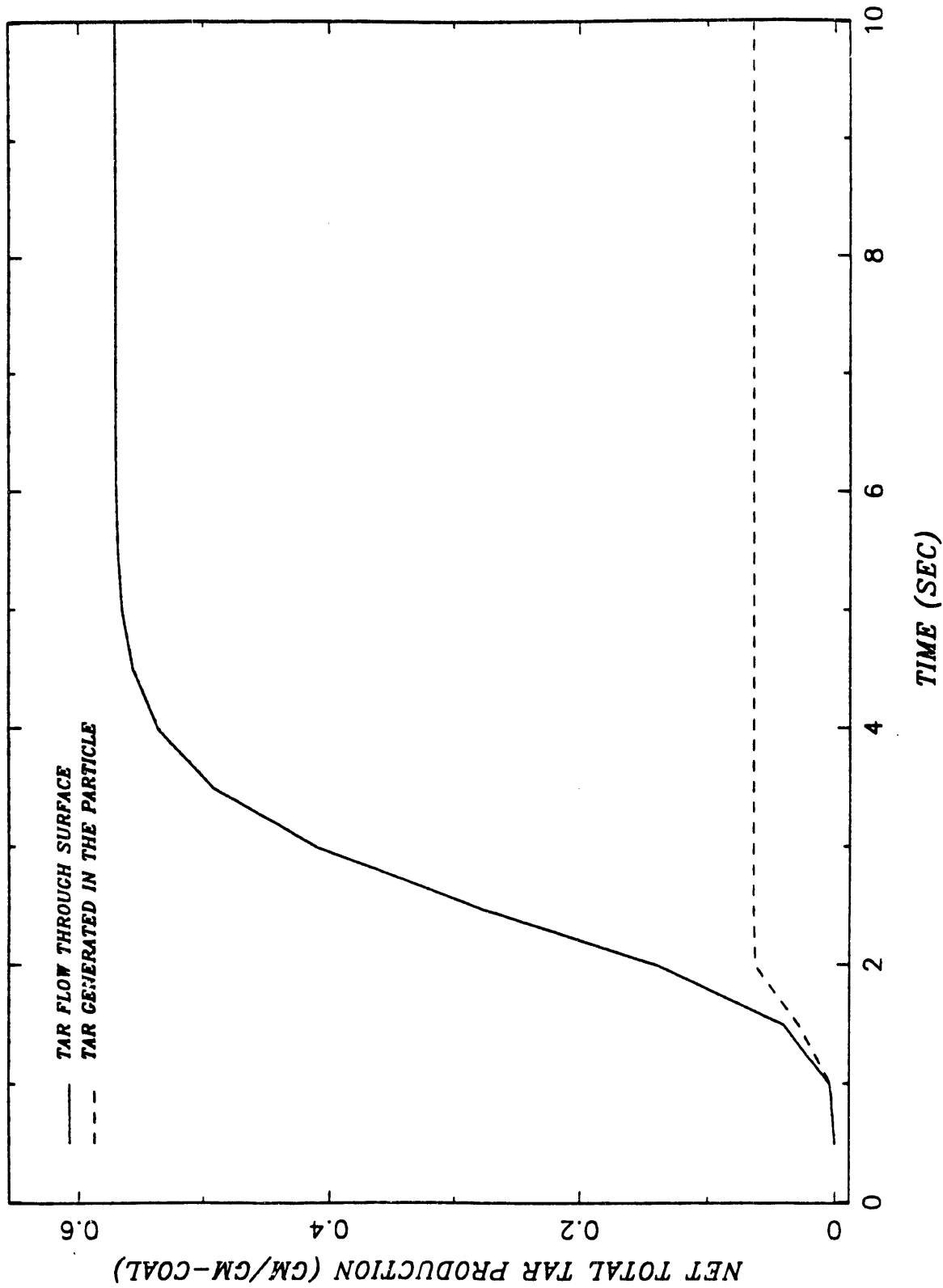


Figure 3.12: Comparison of Total Net Tar Generation for Flow through the Particle Surface Versus Generation Inside the Particle for the Original Wen II Model.

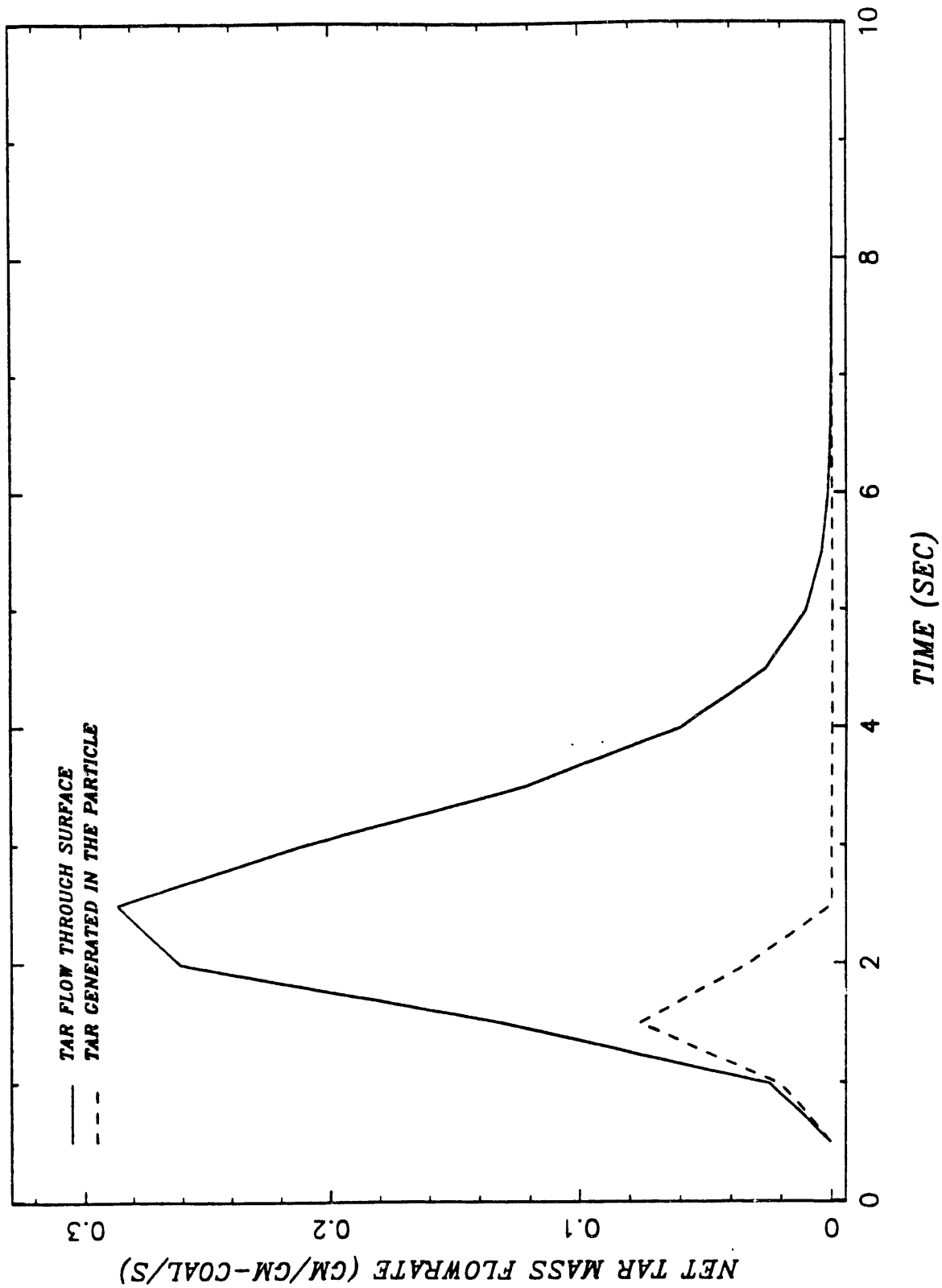


Figure 3.13: Comparison of Instantaneous Net Tar Generation for Flow through the Particle Surface Versus Generation Inside the Particle for the Original Wen II Model.

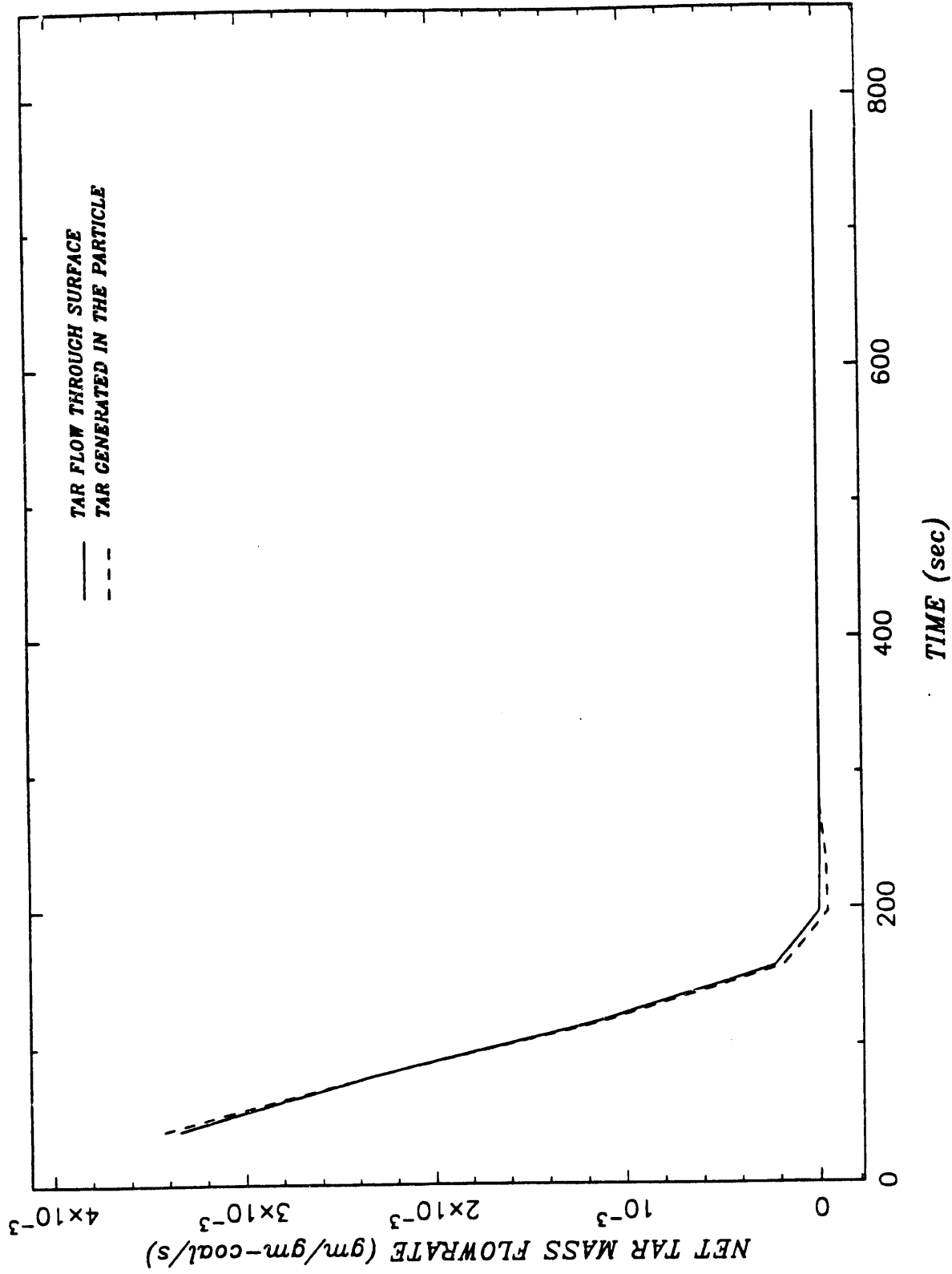


Figure 3.14: Comparison of Total Net Tar Generation for Flow through the Particle Surface Versus Generation Inside the Particle for the Modified Wen II Model.

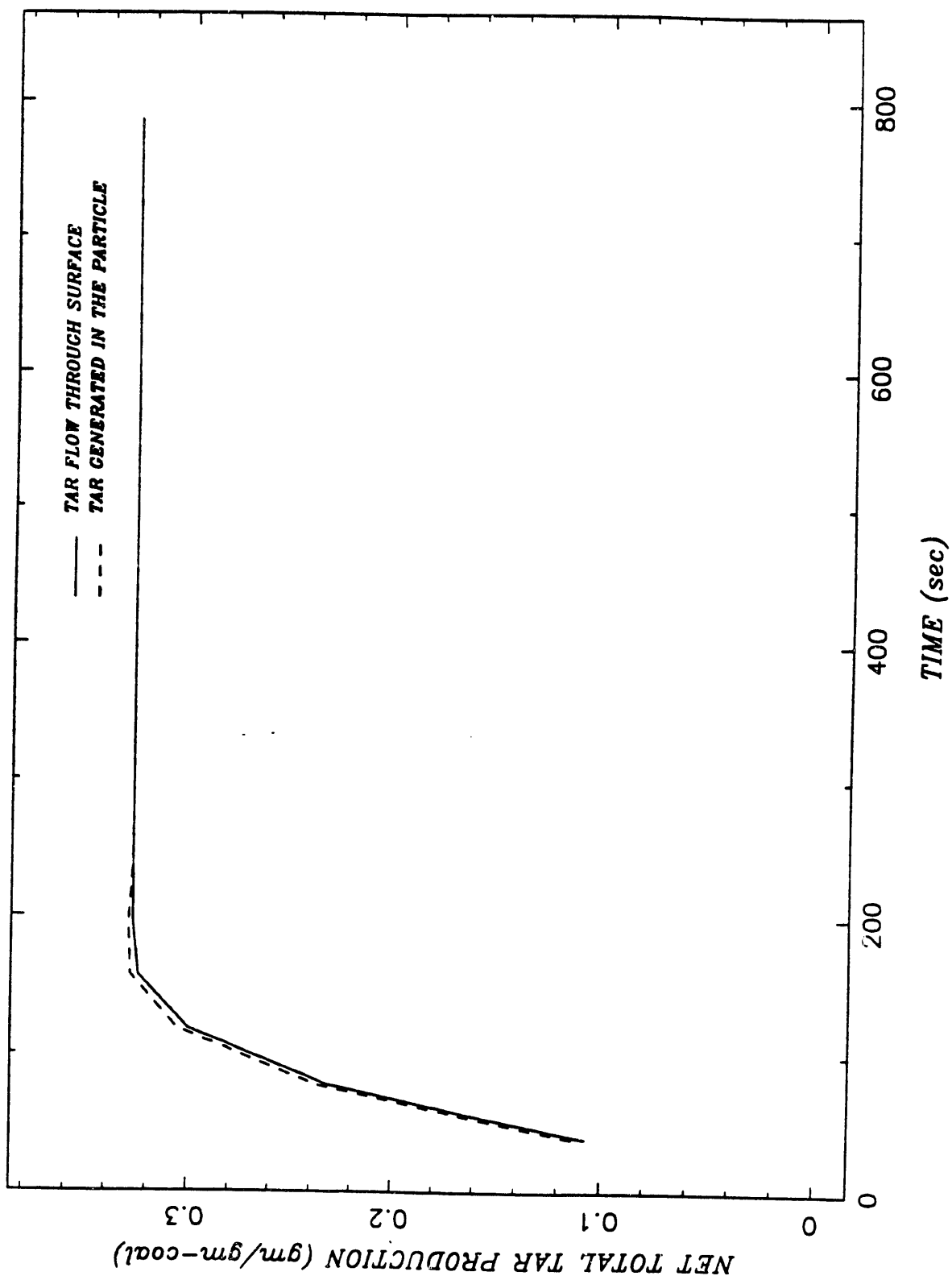


Figure 3.15: Comparison of Instantaneous Net Tar Generation for Flow through the Particle Surface Versus Generation Inside the Particle for the Modified Wen II Model.



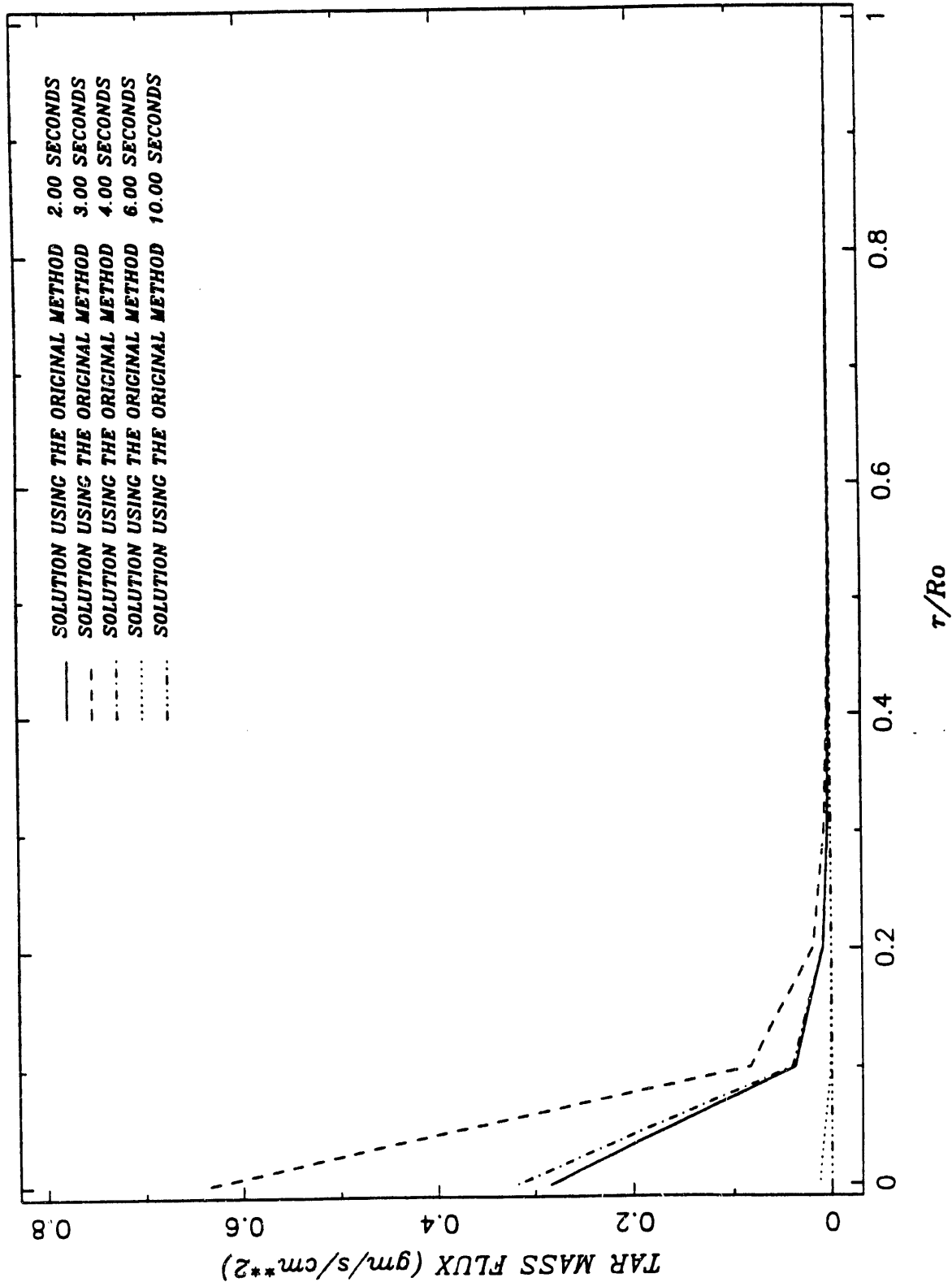


Figure 3.16: Comparison of Tar Mass Flux at Various Times for the Original Wen II Model.

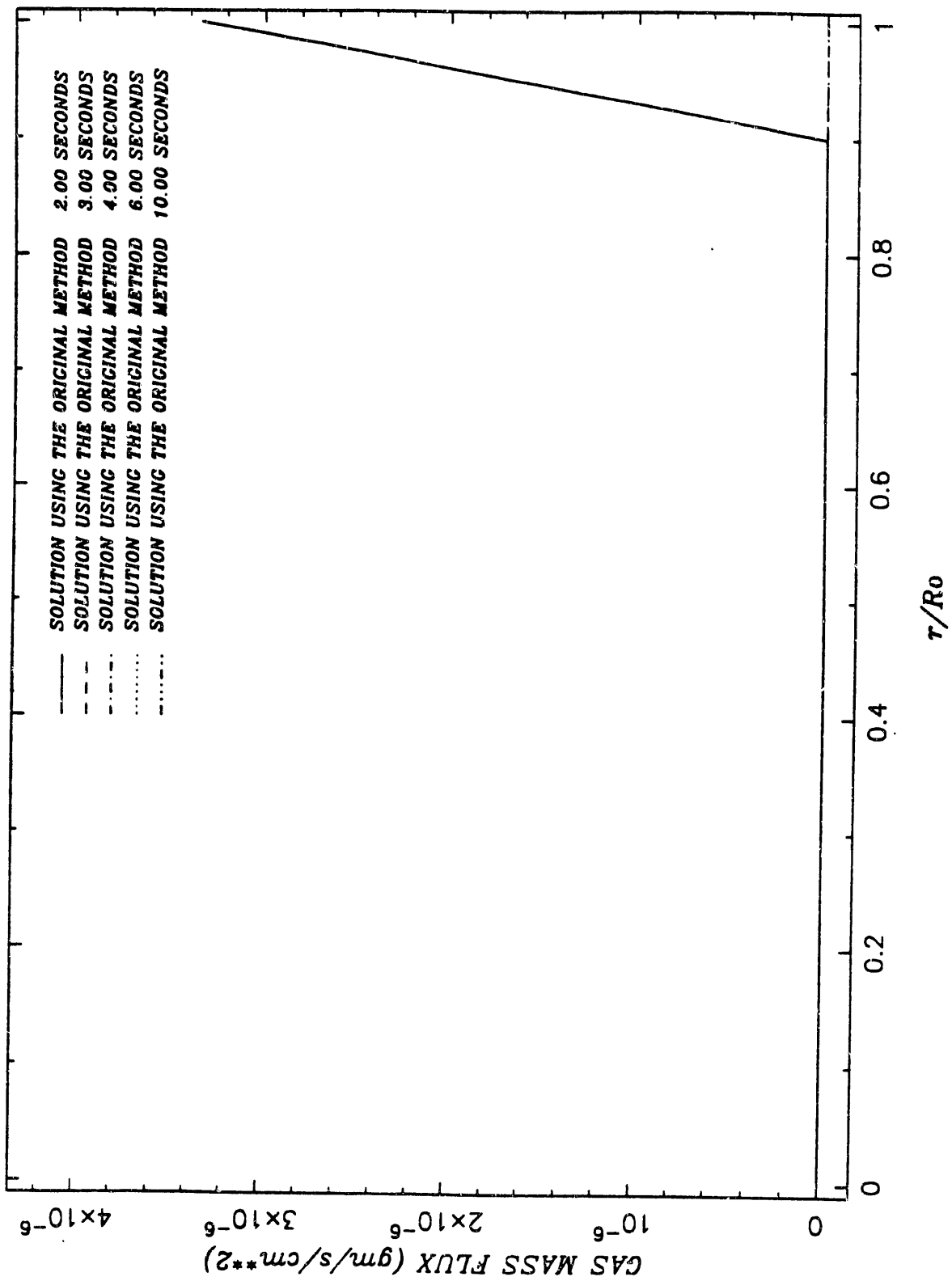


Figure 3.17: Comparison of Gas Mass Flux at Various Times for the Original Wen II Model.

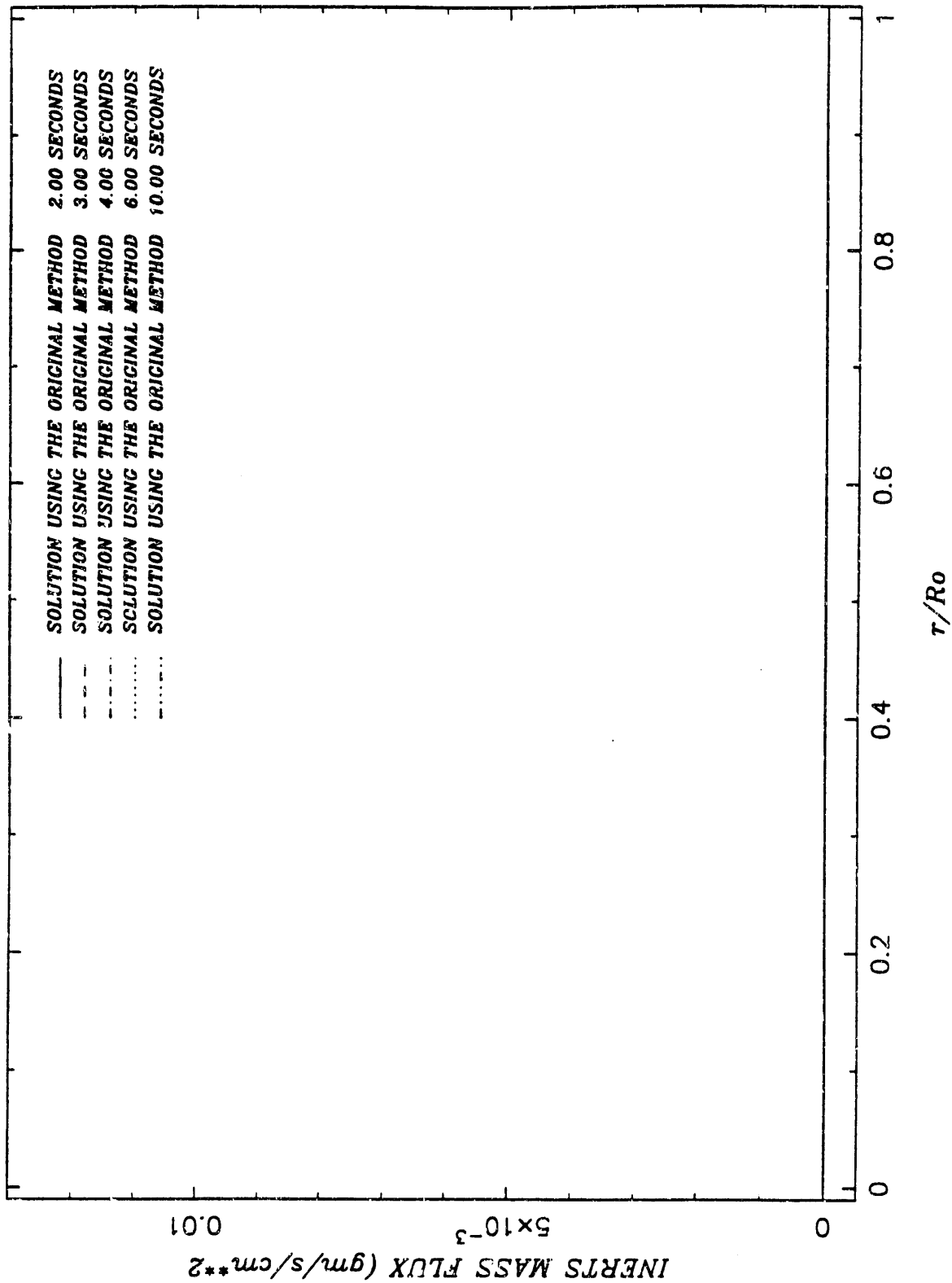


Figure 3.18: Comparison of Inerts Mass Flux at Various Times for the Original Wen II Model.

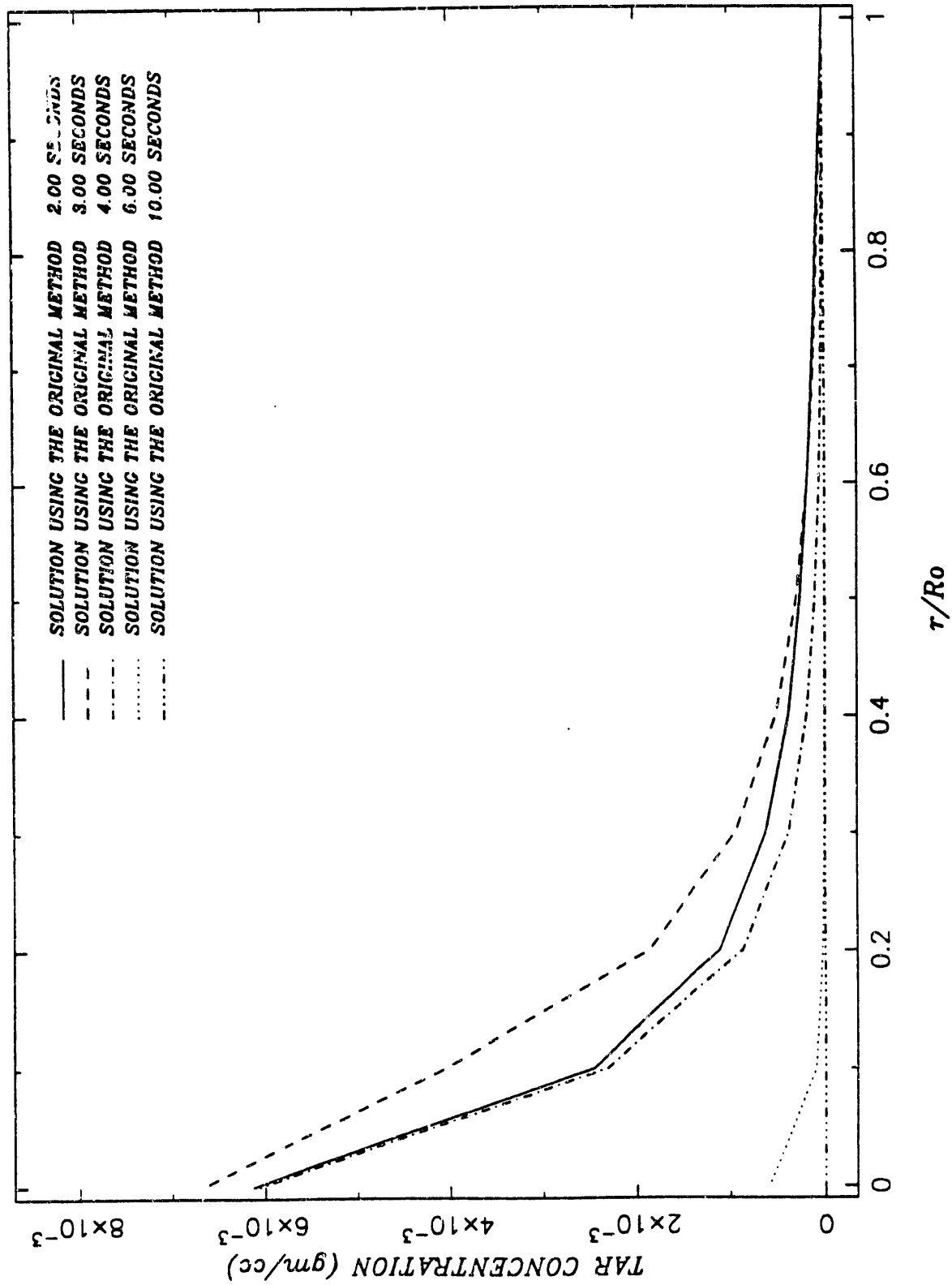


Figure 3.19: Comparison of Tar Concentration at Various Times for the Original Wen Model.

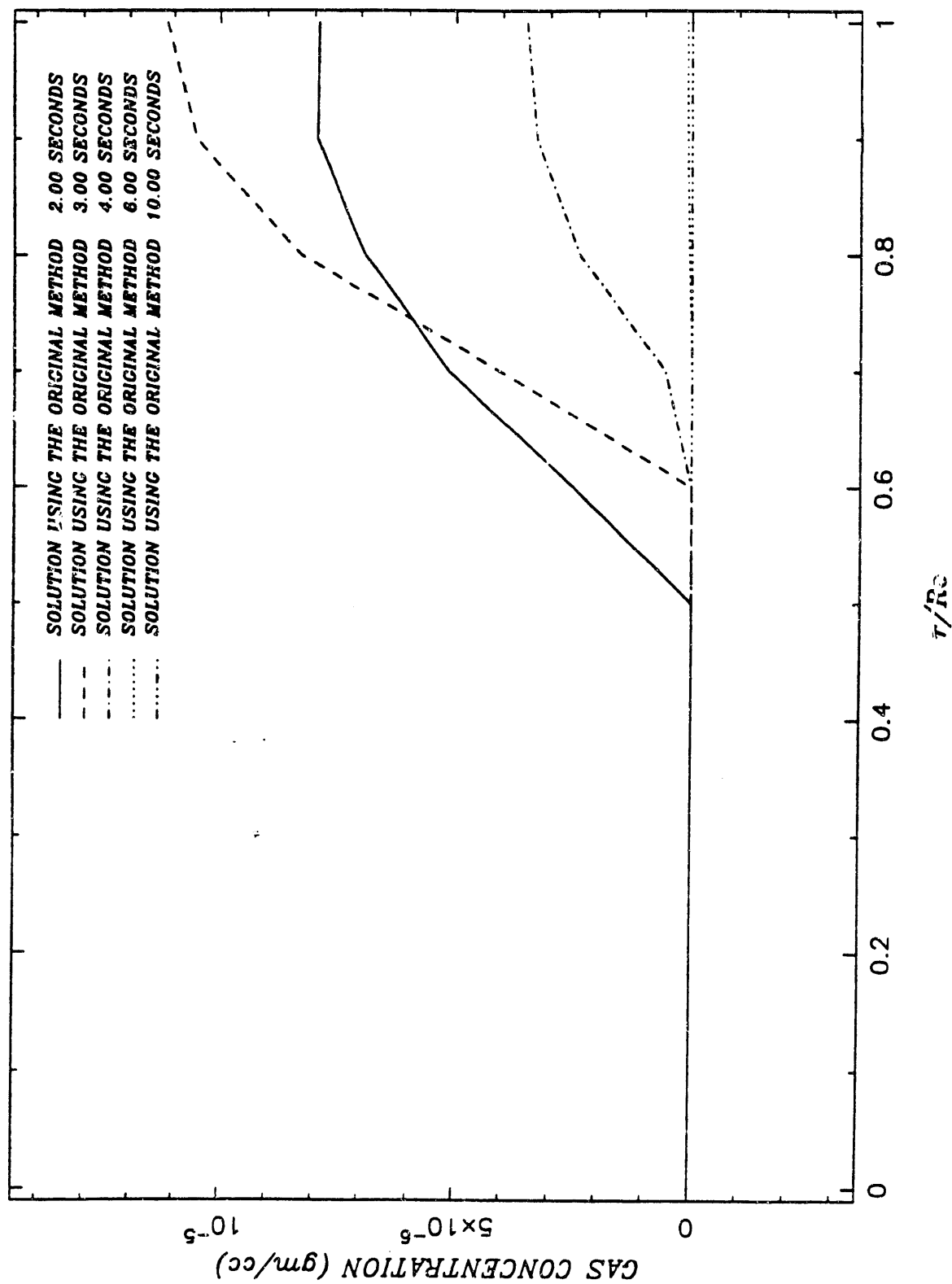


Figure 3.20: Comparison of Gas Concentration at Various Times for the Original Wen II Model.

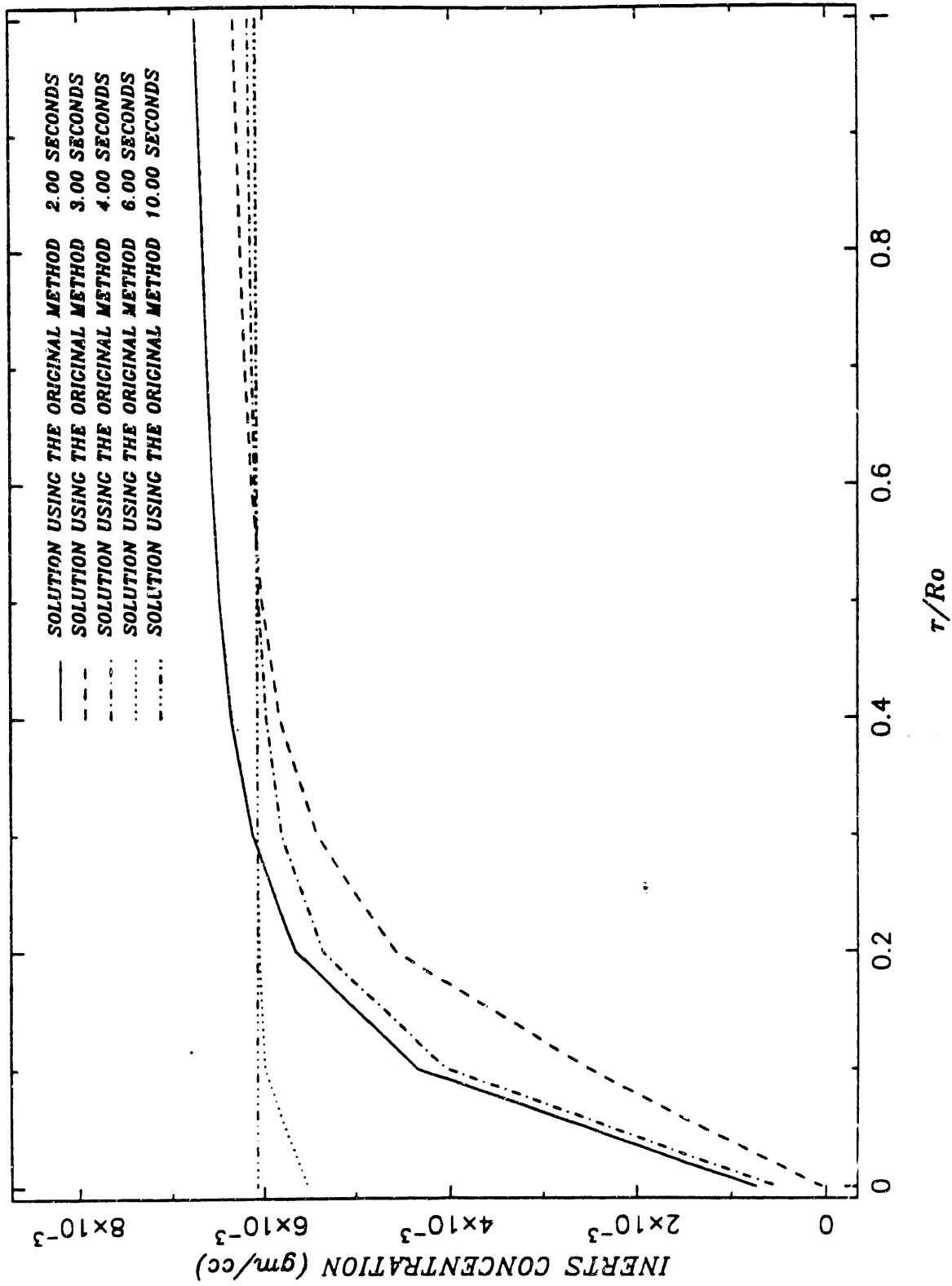


Figure 3.21: Comparison of Inerts Concentration at Various Times for the Original Wen II Model.

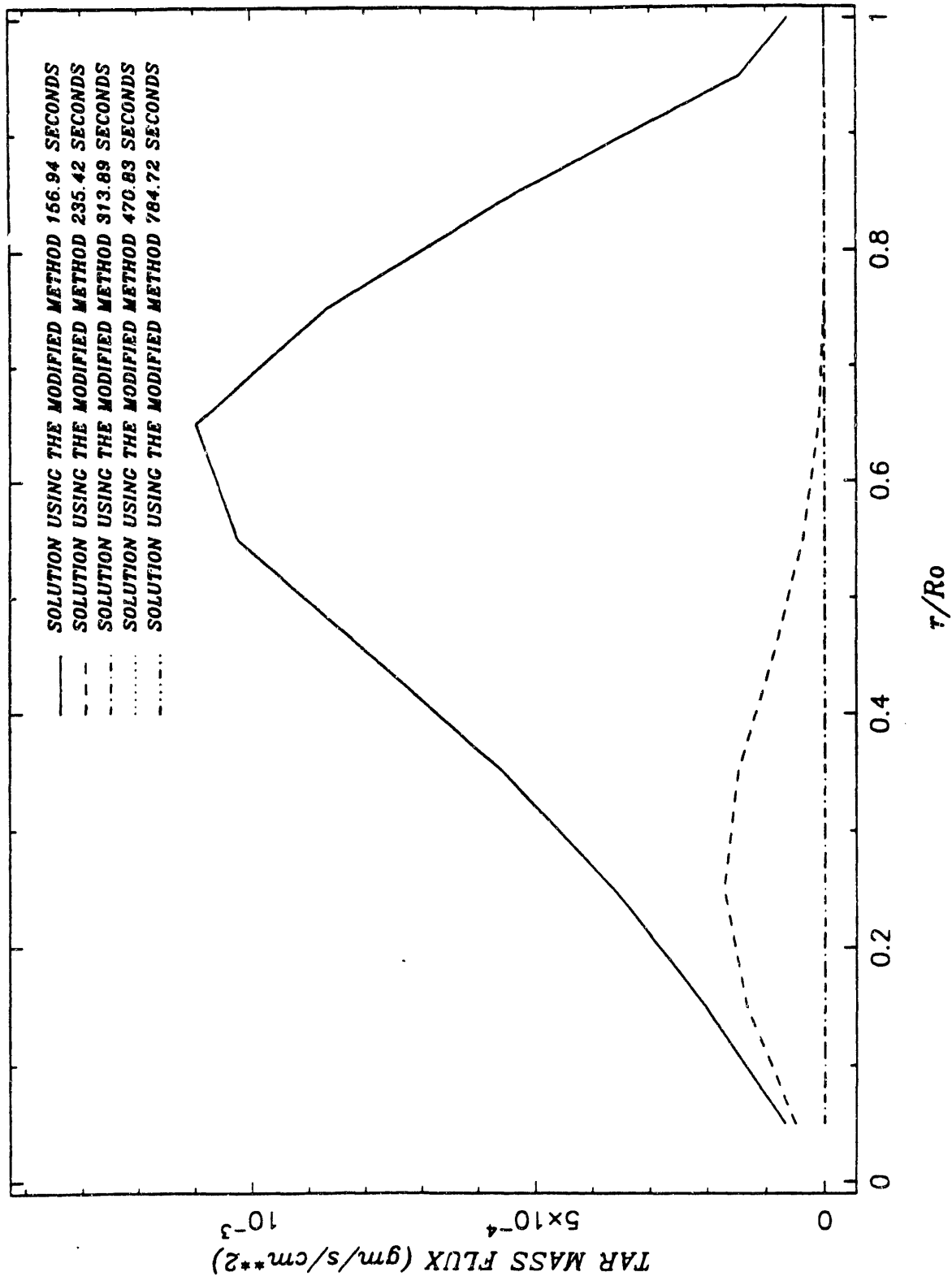


Figure 3.22: Comparison of Tar Mass Flux at Various Times for the Modified Wen II Model.

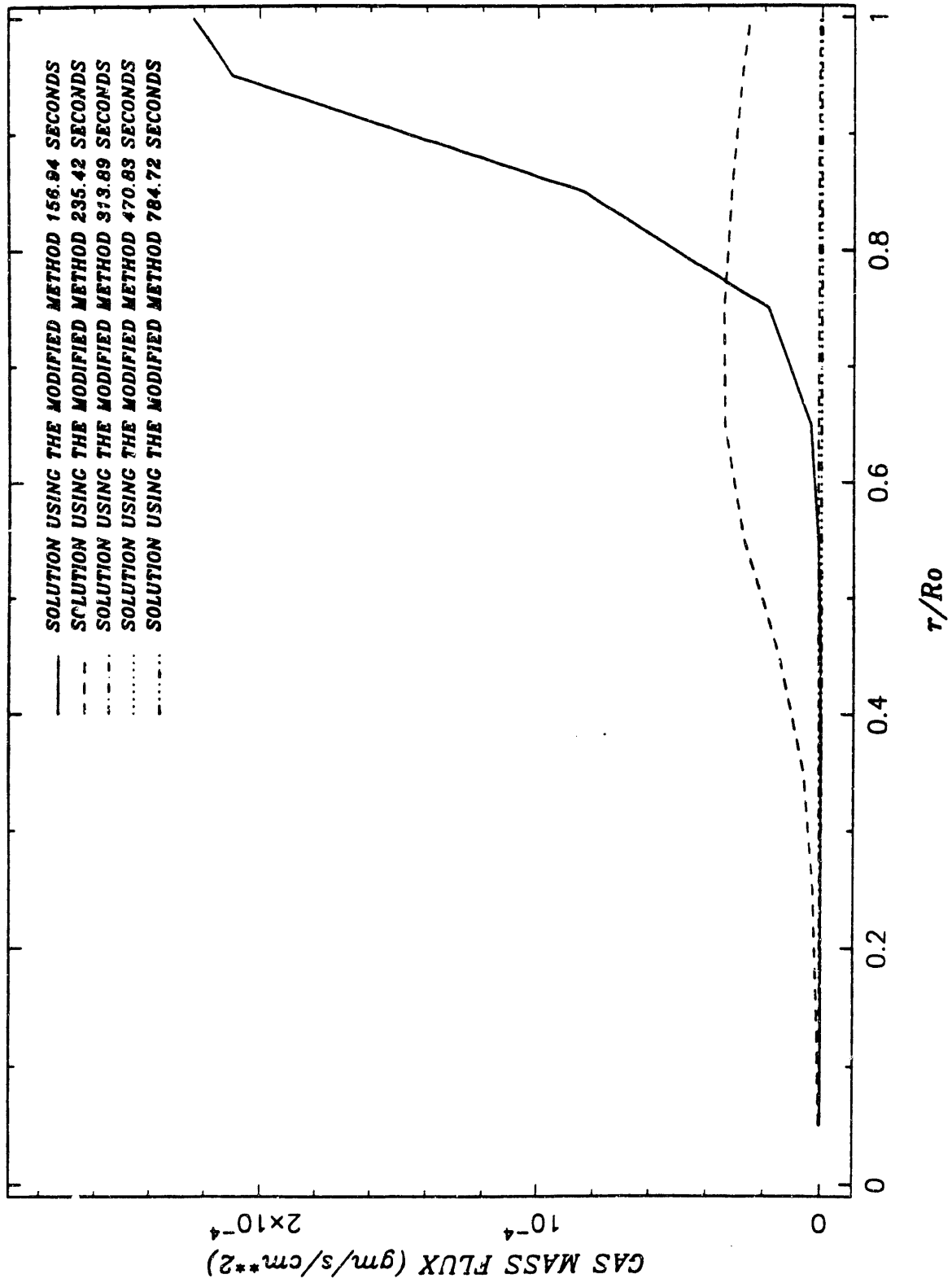


Figure 3.23: Comparison of Gas Mass Flux at Various Times for the Modified Wen II Model.



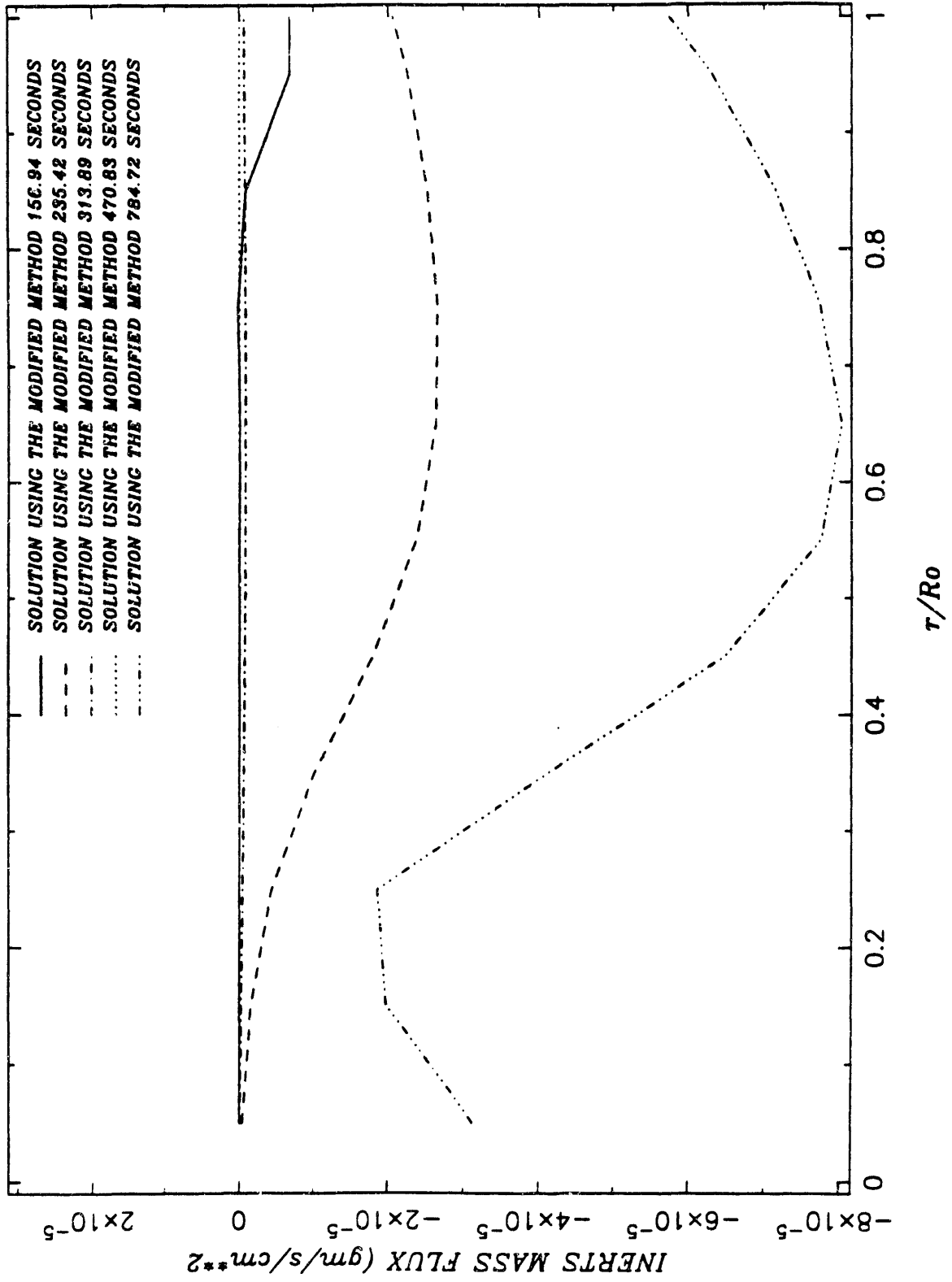


Figure 3.24: Comparison of Inerts Mass Flux at Various Times for Modified Wen II Model.

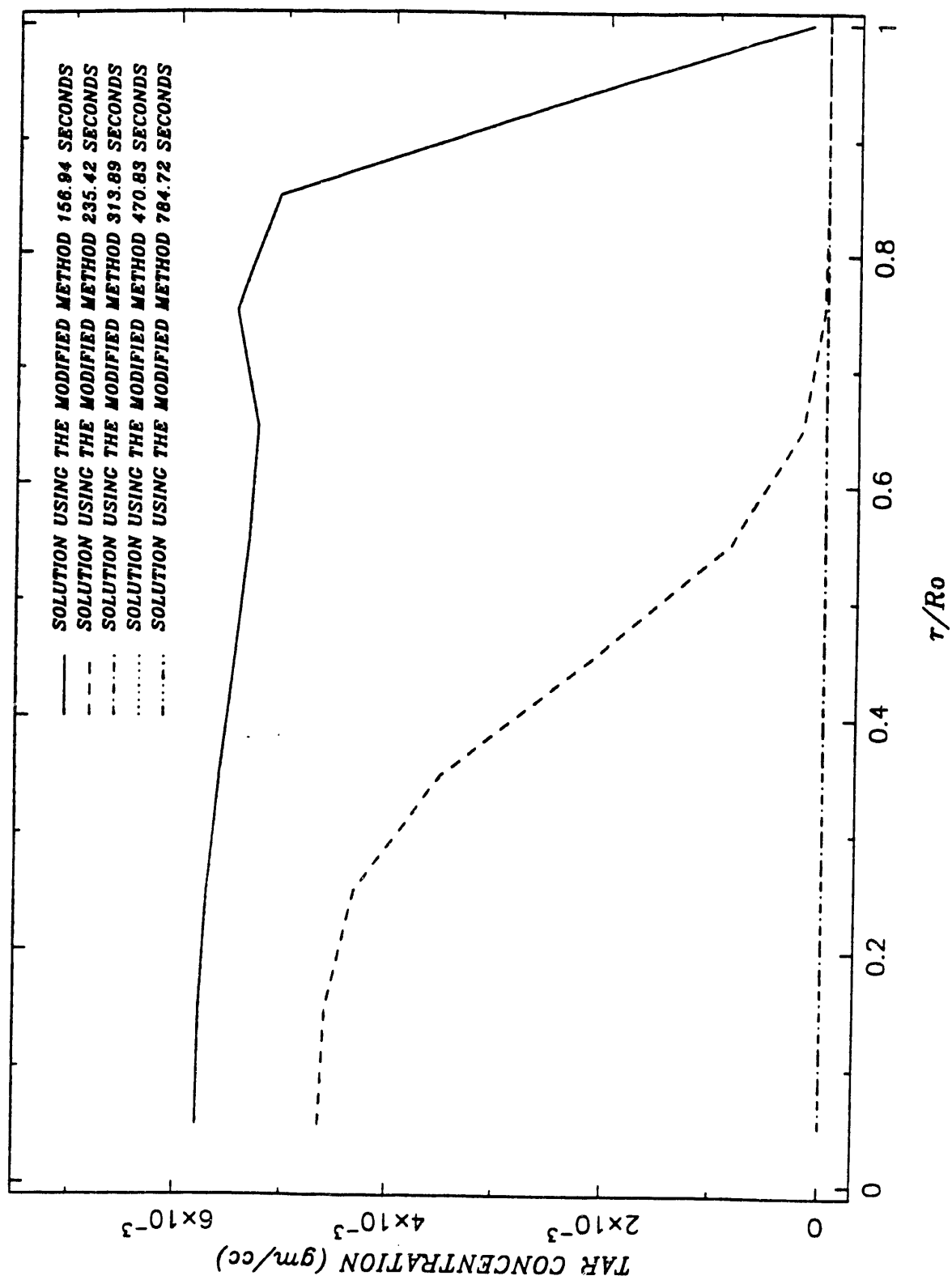


Figure 3.25: Comparison of Tar Concentration at Various Times for the Modified Wen II Model.

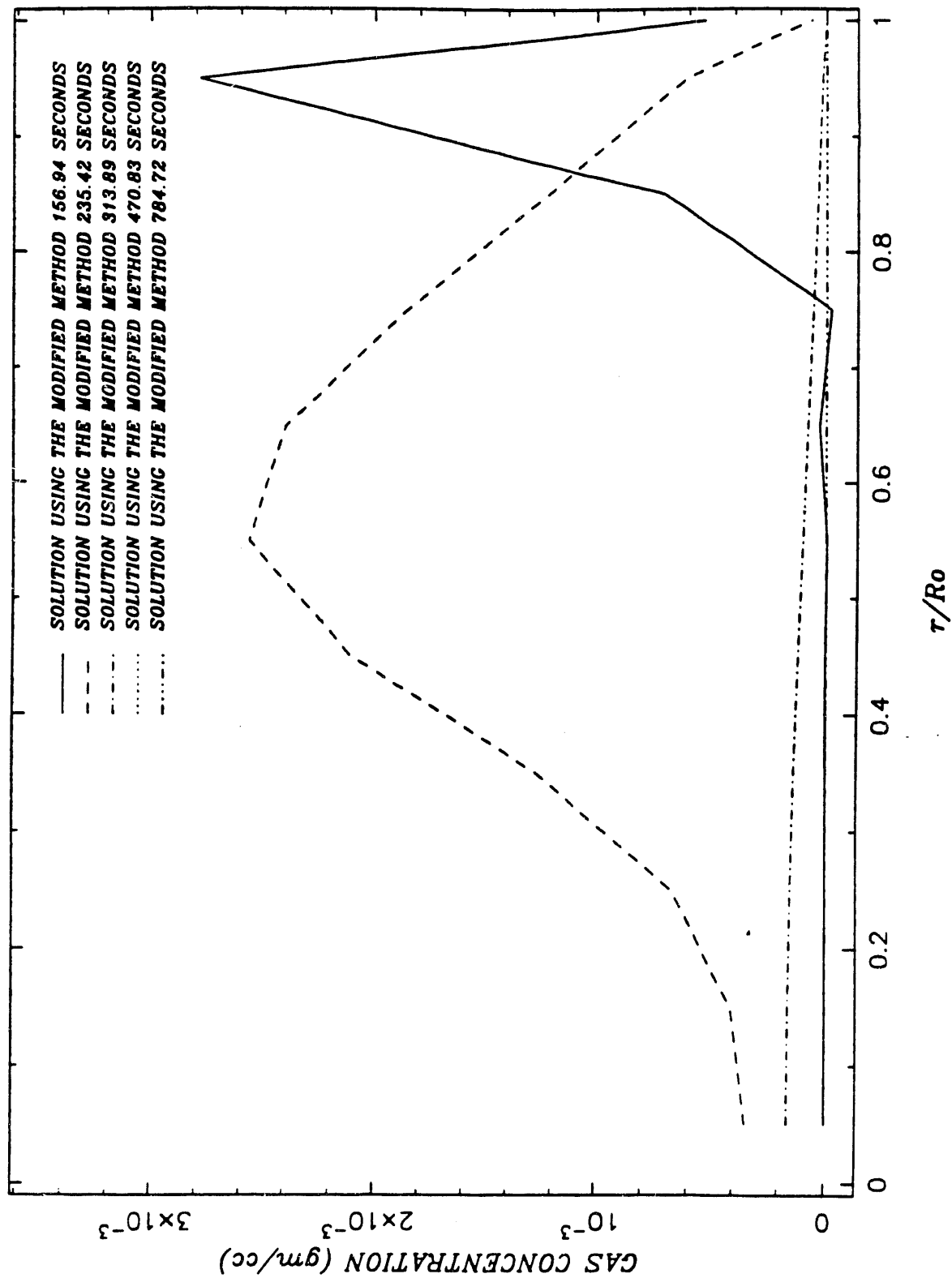


Figure 3.26: Comparison of Gas Concentration at Various Times for the Modified Wen II Model.

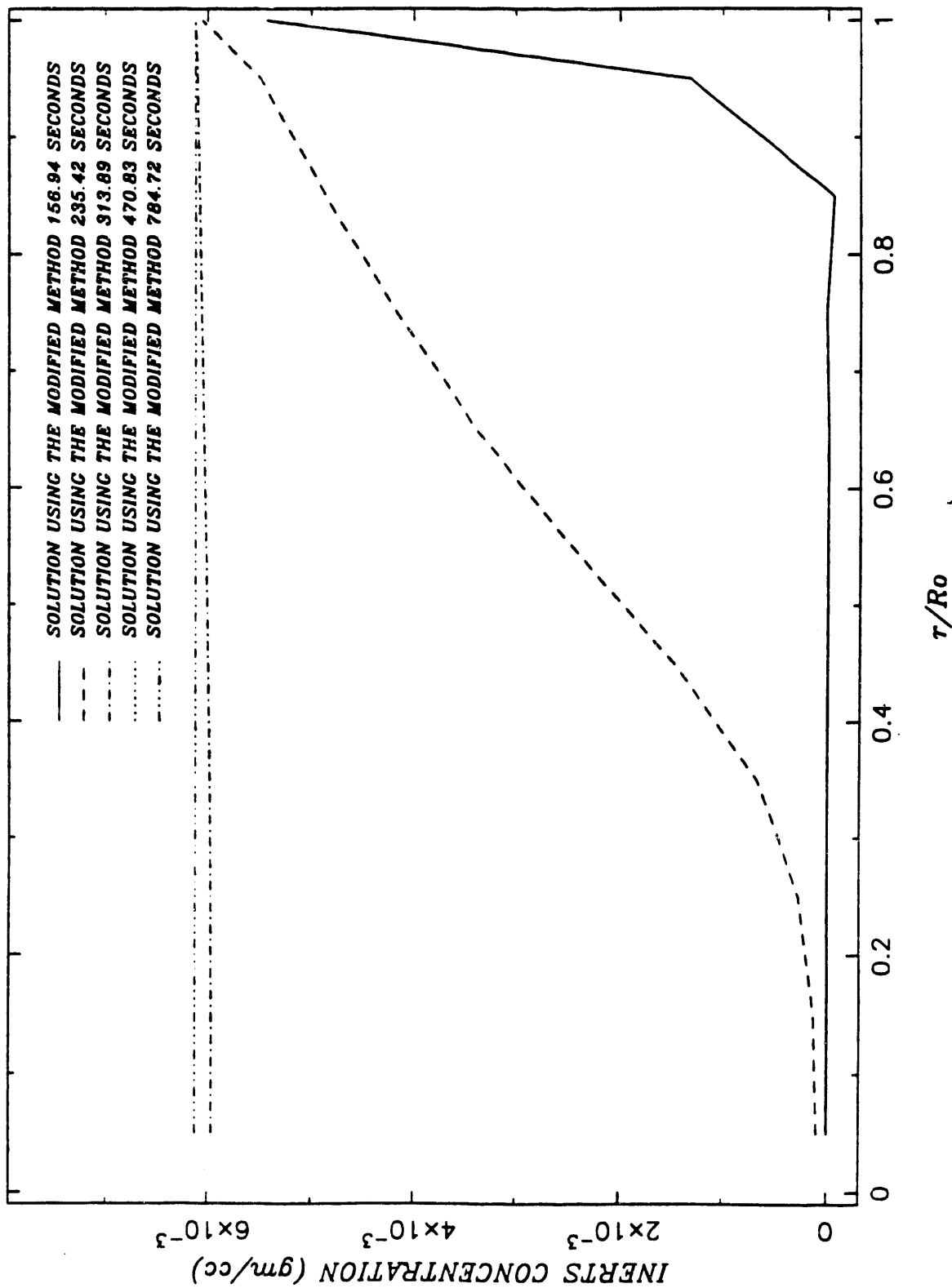


Figure 3.27: Comparison of Inerts Concentration at Various Times for the Modified Wen II Model.

### 3.4.2 GASIFIER PERFORMANCE

In Cases 3 through 12, an air-blown Lurgi Mark IV gasifier was modeled using Illinois #3 as the feed coal. These cases were run as a parametric study on the effect of feed coal size, gasifier bed height, air inlet temperature, and steam inlet temperature. Table 3.3 lists the common input data for Cases 3 to 12, specific to a Lurgi Mark IV and Illinois #6 coal.

**Table 3.3 Coal Input Data for Cases 3 through 12**

Coal Feed Rate, lbs/hr	24981.07				
Proximate Analysis of Illinois #6 Coal, Wt %					
Moisture	10.23	Volatile matter	34.70		
Ash	9.91	Fixed Carbon	45.97		
Ultimate Analysis of Illinois #6 Coal, Wt %					
C	64.2	H <sub>2</sub>	4.3	O <sub>2</sub>	8.1
N <sub>2</sub>	1.2	S	2.8	H <sub>2</sub> O	10.23
Ash	9.10				
Heating Value of Coal, Btu/lb-wet coal	12397.00				
Oxygen Content, mole ratio	0.21				
Temperature of Coal, °F	70.00				
Temperature of Wall, °F	700.00				
Pressure, psig	300.00				
Bed Diameter, ft	12.40				
Heat Transfer Coefficient, Btu/(ft <sup>2</sup> · hr · °R)	17.00				
Bed Voidage	0.40				

In Cases 3 to 7, the input parameters for steam feed rate in Table 3.4 were provided by our sponsor, CRS Serrine, specific to their IGCC system. The inlet temperature of the steam is the saturation temperature at 300 psig, and the inlet air temperature was determined as if the air were compressed from atmospheric pressure to 300 psig with no intercooling. Table 3.4 list the results of this parametric case study.

In Table 3.5 it is worth noting that the size of the coal particle has a direct impact on the amount of tar produced. The larger particles of coal produced less tar than the smaller

**Table 3.4 Input Data for Cases 3 through 7**

Steam Feed Rate, lbs/hr	36472.36				
Air Feed Rate, lbs/hr	48713.09				
Temperature of Steam, °F	420.00				
Temperature of Air, °F	900.00				
PARAMETERS	CASE NUMBER				
	3	4	5	6	7
Initial Coal Particle Size, in	0.50	1.00	1.50	1.00	1.00
Bed Height, ft	8.5	8.5	8.5	7.0	10.00

ones. A reason for this effect is that the larger coal particles take longer to heat up, and more tar cracking occurs. Also, the size of the coal particles affects the heating value of the gas. The larger particles increased the heating value of the coalgas as an effect of the reduced tar production. However, the larger particles reduced the amount of carbon conversion for the gasifier. The larger particles take longer to heat up to the gasification temperatures, allowing less time for the gasification process to occur. The larger coal particles reduce the effective height of the gasifier. Table 3.5 shows how the column height needed for devolatilization depends on the particle size. The additional time for the larger particles to devolatilize reduces the effective height of the gasifier. The devolatilization time will also be affected by the amount of moisture in the coal; i.e. wet coal will take longer to heat up than will dry coal.

The bed height affects the amount of carbon converted to coalgas and the amount of tar produced. The larger bed heights give the coal a longer time to react with the steam and air to produce the coalgas, increasing the carbon conversion efficiency. The amount of tar produced is related to the temperature in the devolatilization zone. For higher beds, the gas has more time to cool before reaching this zone, causing less tar to be cracked into gas.

In Cases 8 to 12, the input data was modified to simulate water injection into the air stream. The effect of the water injection is to cool the inlet air/steam mixture into the bottom of the gasifier. However, the inlet temperature can not be lowered below the dew point. In Cases 8 to 12 the inlet temperature is about 10 °F above the minimum allowable

**Table 3.5 Comparison of Results Cases 3 through 7**

COMPARISON PARAMETERS	CASE NUMBER				
	3	4	5	6	7
Product Gas Composition, mole %					
CO	9.78	10.90	11.80	11.40	10.46
CO <sub>2</sub>	8.92	9.13	8.97	8.69	9.47
H <sub>2</sub>	18.36	20.66	21.81	20.42	20.77
CH <sub>4</sub>	0.07	0.06	0.05	0.05	0.07
H <sub>2</sub> O	34.34	31.43	29.99	31.69	31.32
N <sub>2</sub>	28.08	27.38	26.94	27.31	27.46
H <sub>2</sub> S	0.46	0.44	0.44	0.44	0.45
Amount of Tar, lbs/hr	5444.84	4224.20	3277.89	4002.66	4420.93
Temperature of Exit Gas, °F	1068.15	1108.92	1148.71	1137.76	1089.25
Maximum Temperature, °F	1964.03	1961.63	1946.60	1958.75	1964.06
Carbon Conversion, %	95.72	95.96	94.93	95.15	96.45
Heating Value of Dry Gas, Btu/SCF	143.16	151.19	156.27	152.84	149.62
Heating Value of Wet Gas, Btu/SCF	94.70	104.29	109.80	104.91	103.03
Heat Loss, 10 <sup>6</sup> Btu	4.11	3.81	3.40	3.15	4.45
Devolatilization Length, in	2.96	10.14	18.98	10.05	10.43
Devolatilization Time, min	4.47	15.31	28.64	15.17	15.74

temperature at 300 psig at which the air would be saturated. These cases were run making no additional changes to the input data in Cases 3 to 7. The steam flowrates were adjusted until the maximum temperature within the gasifier in this run was same as that of Case 4. The air flow rate was adjusted until the carbon conversion was also nearly the same. The adjusted values appear in Table 3.6 as the input data for Cases 8 to 12. The primary purpose of Cases 8 to 12 was to investigate the possibility of reducing the amount of steam introduced into the system.

As seen in comparing Table 3.7 with Table ??, the results indicate that water injection does decrease the steam requirements, and also increases the heating value of the wet gas. The carbon conversion can be increased by increasing the air flow.

**Table 3.6 Input Data for Cases 8 through 12**

Steam Feed Rate, lbs/hr						27479.18
Air Feed Rate, lbs/hr						51960.63
Temperature of Steam, °F						365.00
Temperature of Air, °F						365.00
	CASE NUMBER					
PARAMETERS	8	9	10	11	12	
Initial Coal Particle Size, in	0.50	1.00	1.50	1.00		1.00
Bed Height, ft	8.5	8.5	8.5	7.0		10.00

### 3.5 Suitability of Model for Predicting Lurgi Gasifier Performance

In this section we shall discuss some of the limitations of the current computer model with regards to its ability to provide accurate and relevant information needed to design an IGCC power plant. We shall begin by reviewing some specific information about a commercial fixed bed coal gasifier, known as the Lurgi Gasifier. An air blown Lurgi will probably be employed in the forthcoming IGCC demonstration plant. By reviewing the design of this gasifier, we wish to better define the requirements of the computer model and to point out potential problem areas which our computer model will not be capable of analyzing.

The Lurgi Gasifier reportedly [10 - 13] will take all types of coal feedstock, but requires an agitator and/or increased steam rate for strongly caking coals. The size of the coal is normally from 0.125 to 2.0 inches in diameter with up to 10% coal fines. The coal feed rate is in the range of 100 to 400 lb/hr-ft<sup>2</sup>. Crushing and sizing may be required depending on the coal, and drying may be required to reduce the moisture content. Partial oxidation of the feed coal may also be required for strongly caking coals.

In oxygen-blown operation, 1.01 to 3.24 kg of steam per kg of coal are required, while as little as 0.6 kg of steam/kg- coal are required for air-blown operation. The gasifier requires from 0.23 to 0.61 kg of pure oxygen/kg-coal or from 1.3 to 1.9 kg of air/kg-coal for operation.



**Table 3.7 Comparison of Results Cases 8 through 12**

COMPARISON PARAMETERS	CASE NUMBER				
	8	9	10	11	12
Product Gas Composition, mole %					
CO	11.32	12.65	13.59	13.13	12.17
CO <sub>2</sub>	9.24	9.43	9.31	9.03	9.76
H <sub>2</sub>	18.68	21.31	22.67	21.15	21.34
CH <sub>4</sub>	0.08	0.07	0.06	0.05	0.08
H <sub>2</sub> O	27.53	24.33	22.75	24.52	24.32
N <sub>2</sub>	32.66	31.72	31.14	31.63	31.83
H <sub>2</sub> S	0.50	0.48	0.47	0.48	0.49
Amount of Tar, lbs/hr	5289.96	4051.60	3115.67	3835.77	4253.83
Temperature of Exit Gas, °F	1056.18	1100.09	1141.47	1123.14	1079.46
Maximum Temperature, °F	1990.97	1991.29	1986.60	1995.11	1986.41
Carbon Conversion, %	95.45	96.08	95.37	95.44	96.50
Heating Value of Dry Gas, Btu/SCF	139.03	148.04	153.30	149.55	146.36
Heating Value of Wet Gas, Btu/SCF	101.33	112.42	118.72	113.26	111.19
Heat Loss, 10 <sup>6</sup> Btu	4.15	3.81	3.34	3.14	4.50

The coal residence time in the gasifier is approximately one hour. The operating pressure varies from 2.1 to 3.2 MPa with normal operating conditions of 2.1 MPa for air-blown and 3.0 MPa for oxygen-blown. The gas outlet temperature varies from 644 to 866 °K with a normal gas exit temperature of 744 °K [1].

The Lurgi Gasifier reportedly has many advantages. It can accept either caking or non-caking coals and use either oxygen or air as the oxidation media. The high pressure of the Lurgi favors the formation of methane in the gasifier, and high pressure is very advantageous for IGCC applications. The gasifier is highly developed and has been operated for many years.

On the other hand, the Lurgi Gasifier also has some disadvantages which become major concerns when we consider it for use in an IGCC system. One concern is the effect of the by-product tars and oils on the hot gas cleanup system, on gas turbine parts, and on components used for handling the by-products. The tars, which contain nitrogen, are also expected to contribute significantly to air pollution. The computer model described above does predict the amounts of tar produced depending on the coal properties and the gasifier operating

conditions. However, the model predictions have not been checked against experimental data. We expect that the present model will have very limited value for defining problems due to tar in the product gas.

A second concern is the ability of the gasifier to operate with caking coals. It is known that the coal feed rate must be reduced for caking coals and that an agitator may be required. However we are not able to predict quantitatively how the swelling index of the coal affects the gasifier's performance. Our computer model does not even consider the swelling index of the coal.

The Lurgi Gasifier also has problems processing coal fines. The fines are swept out of the gasifier with the product gas and also tend to block the flow of gases through the bed of the gasifier, particularly when caking coals are used. Our computer model does not account for fines in the coal. It assumes a constant coal particle diameter and uniform flow through the gasifier bed.

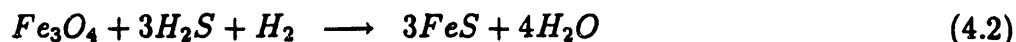
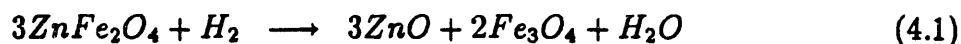
The discussion above helps to define the way that the present computer model may be used in the IGCC system design study. Computer models are becoming increasingly useful for systems evaluation, but it is recognized that they have limitations. Operating experience often reveals the importance of simplifying assumptions in the models. In the case of the IGCC system, it is probable that coals will have to be selected so that caking is not a problem, and sized so that fines are kept at an acceptable level. The overall system design will have to be such that it can accommodate tars in the product gas. The computer models may still be valuable even though they do not account for some very important effects. However, the current gasifier model needs to be improved in several important areas. The devolatilization model needs updating. New rate equations should be programmed into the model so that it predicts quantities of methane and tar accurately. The model should account for more than one coal particle diameter.

## 4. Hot Gas Cleanup Model

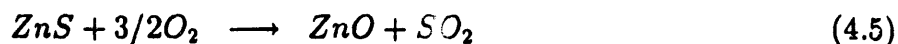
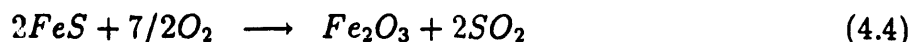
### 4.1 Desulfurization Background

Zinc ferrite is formed by heating a mixture of zinc and iron oxide. Both zinc oxide and iron oxide have long histories of removing hydrogen sulfide from gas streams. Alone, iron oxide works well for hydrogen sulfide concentration above 150 parts per million, and is very easy to regenerate [14,15]. Zinc oxide by itself can perform to as low as 2 parts per million [14,16] but is more difficult to regenerate. However, together they combine the high sulfur affinity of zinc oxide with the good regenerability of iron oxide [14,17]. Zinc ferrite is a primary candidate for hot gas desulfurization in the integrated gasification combined cycle system [18]. The following are the governing chemical reactions:

Sulfidation,



Regeneration,



It is important to operate zinc ferrite at the proper temperatures. Although the sulfidation temperature limitations are not agreed upon exactly, the general consensus is that the sorbent is most effective between 1000 °F and 1200 °F [14,19]. Within this range, magnetite ( $Fe_3O_4$ ) is formed from the zinc ferrite and the sulfidation continues very well. Outside of the prescribed temperature range, species other than magnetite (iron/iron carbides or wustite) will be formed in the absorption bed, creating adverse effects. The sulfidation reaction is slightly exothermic and therefore will cause a rise in temperatures.

The regeneration is done by mixing air with the sorbent. This process is extremely exothermal and may be the cause of problems [14,19]. If the reaction is performed with pure air (21 % oxygen), temperatures may reach as high as 1700 °F. This is unacceptable because temperatures above 1500 °F will cause sintering of the sorbent, resulting in the loss of reactivity. This will render the sorbent potentially unusable for multi-cycles. There also exists a minimal temperature below which other unwanted process take place. At temperatures below 1200 °F considerable sulfate formation will occur. If the sulfate is not removed, a sulfidation cycle will start which emits high sulfur levels and contaminates the absorption cycle.

For the above reasons it is important to restrict the temperatures within 1200 °F and 1500 °F. This can be done by applying strict control to the stream of gases that react with the sorbent during the regeneration process. Inlet temperature, mass flow rate, and oxygen mole fraction of the reacting gases affect the temperatures in the regeneration cycle. Thus, these variables can be controlled to keep the reaction temperature within usable limits.

#### 4.2 Regeneration Cycle System Design

The General Electric Company has developed [20,21] a system to provide oxygen in controlled amounts for proper regeneration of zinc ferrite. Figure 4.1 is a schematic of the system.

The General Electric design involves a three zone chamber. In the first zone 30 percent of the oxidation reaction that regenerates the sorbent is expected to occur. Oxygen levels in this zone start at 1 percent when the sorbent has been loaded, and then increases to about 2.5 percent to keep the reaction at 1250 °F. Although there is considerable sulfate formation in this zone, higher temperatures in the following zone will remove the sulfate. Gas inlet temperatures in this area range from 750 °F to 900 °F.

The second zone is where the completion of the reaction takes place. In this region a gas stream with 4 to 5 percent oxygen is input, allowing the reaction temperatures to be

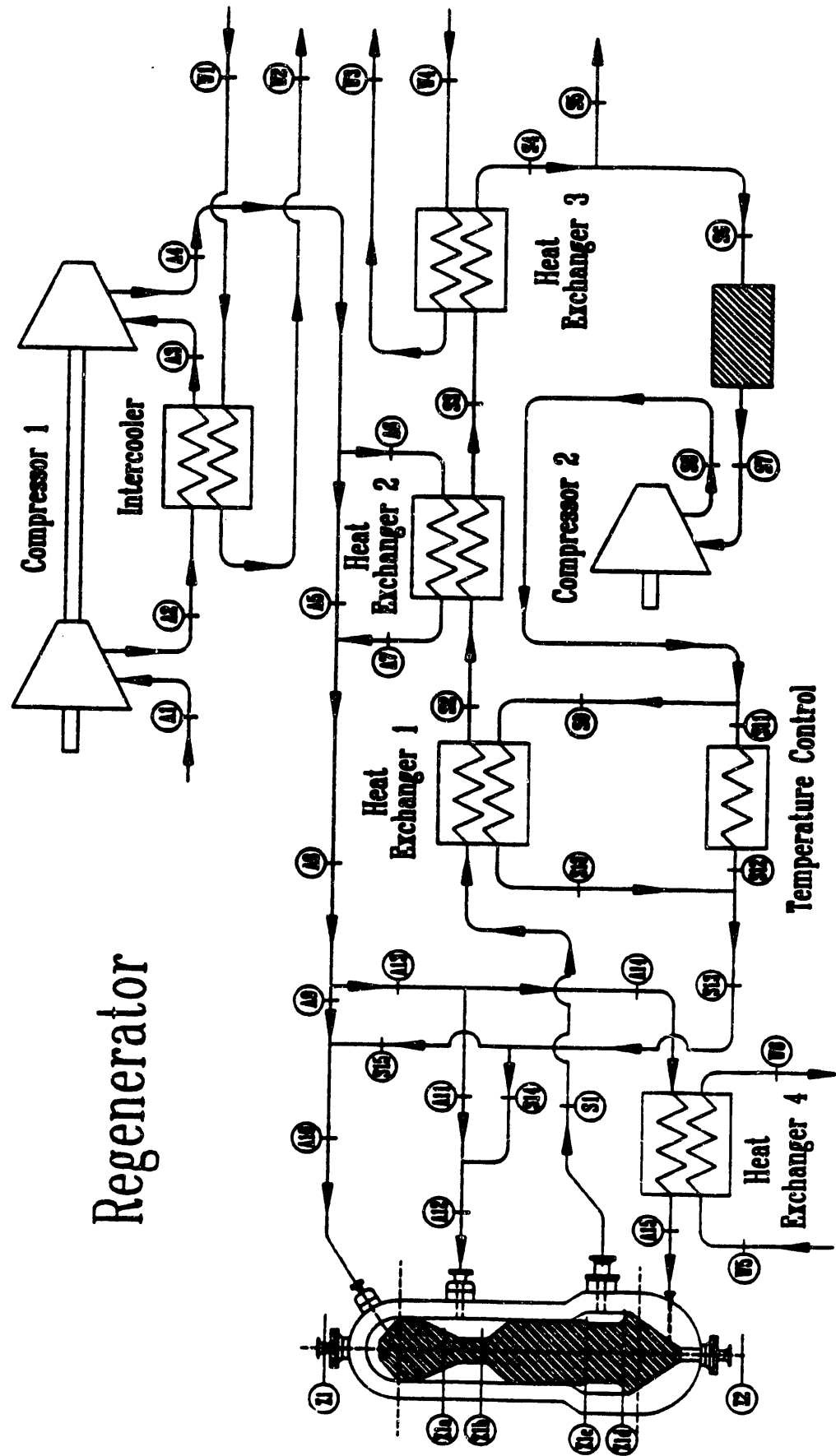


Figure 4.1 Closed Loop General Electric Regeneration Subsystem Design

kept at 1450 °F. This allows any sulfates created in the first zone to be destroyed without sintering the sorbent. Gas inlet temperatures are also kept between 750 °F and 900 °F in this zone. The flow rates are controlled so as to keep the excess oxygen to under 1 percent after exiting this area.

The purpose of the third zone is to cool the regenerated sorbent to between 800 °F and 1000 °F, while ensuring complete regeneration. This is done by introducing air (21 percent oxygen) at 600 °F to 800 °F in the bottom of the reactor, therefore purging any sulfur dioxide from the system. Mass flow rates in this area are controlled to ensure the needed sorbent exiting temperature. The following is a summary of inlet stream conditions required for the GE regeneration cycle.

**Table 4.1 Inlet Conditions for GE Regeneration Cycle**

	Percent Oxygen Inlet	Gas Inlet	Reaction @
Zone 1	1 - 2.5%	750 - 900 F	1250 F
Zone 2	4 - 5%	750 - 900 F	1450 F
Zone 3	21%	600 - 800 F	N / A

The flow system in Figure 4.1 provides the inlet streams at the conditions prescribed in the table. Air containing oxygen for regeneration of the zinc ferrite is compressed to about 20 atmospheres in compressor 1. In an IGCC system this air would probably be bled from the outlet line of the booster compressor so that compressor 1 would not be required. The air is preheated at heat exchanger 2 using waste heat from the exothermic reaction in the regenerator. The air is mixed with recirculating gases to provide oxygen concentrations at the inlet ports as specified in the above table. Heat exchanger 1 preheats the recirculating gases. The temperature controller insures that the gas temperatures remain within the limits prescribed, but is not required under nominal operating conditions. Heat exchanger 3 cools the exhaust gases to the temperatures required for the sulfur recovery process. If a direct sulfur recovery system is employed, heat exchanger 3 is not required. A filter and/or heat exchanger to cool the gases may be required upstream of compressor 2, which is needed to

offset friction in the flow passages. Heat exchanger 4 cools the air entering the final stage of the regenerator.

### 4.3 Computer Model

A computer code was written for analyzing the requirements and performance of the system in Figure 4.1. The program determines flows required in all of the branches of the system, heat exchanger effectiveness requirements, and temperatures. The model incorporates energy balances, overall flow balances, and elemental component balances. It does not calculate pressure losses due to friction. Therefore, the requirements of compressor 2 can only be estimated.

The program is useful for determining how the system requirements depend on variables in the regeneration cycle. One variable is the concentration of  $SO_2$  in the recirculating gases. The optimum concentration may change depending on the design of the sulfur recovery system. A second variable is the amount of sulfur loaded in the zinc ferrite sorbent. This may change with the operating conditions of the gasifier and with the age of the sorbent. Fluid temperatures such as the temperatures at A7 and the temperatures at S5 are variables that can be changed by sizing the heat exchangers or by adjusting the by-pass flow. It is important to know how the system requirements depend on the design variables since the system must function properly over a wide range of operating conditions. The computer model makes it easy to study the effects of changes in the operating conditions.

Parametric analysis described below were made for nominal gasifier operating conditions. In all cases, the flow of sorbent into the regenerator is calculated based on the following gasifier parameters:

**Table 4.2** Conditions for Parametric Analysis

*	Illinois #6
	7,258 lb/hr Sulfur in Coal
	644,108 lb/hr Hot Gas

The input to the model can be easily adjusted to account for variations in the coal gas flow and composition.

#### 4.4 Parametric Analysis

A parametric study was performed to determine the effects of the temperature at A7, the temperature at S5, the  $SO_2$  mole fraction at S5, and the percent of reacted sorbent at the regenerator inlet. Flow rates and/or temperatures were adjusted to ensure that the reactor zone temperatures were kept at 1250 °F (zone 1) and 1450 °F (zone 2).

##### 4.4.1 $SO_2$ Mole Fraction / $O_2$ Mole Fraction at S5

The percentage of  $SO_2$  in the sulfur rich stream became an important parameter for analysis because of the coupled relationship with the oxygen in the loop. From Figure 4.2 we can see that as the amount of oxygen decreases the amount of sulfur dioxide increases. This is due to the fact that sulfur dioxide is composed of oxygen. However, less obvious is the fact that a 13.2% increase in sulfur dioxide (.114 %  $\rightarrow$  .129 %) results in a 63.6% decrease in oxygen (.0360%  $\rightarrow$  .0131%). This relatively large change in oxygen can have a sizable effect on flow rates and required heat exchanger effectiveness.

From Figure 4.3 it can be seen that all the mass flow rates, with the exception of A15, increase as  $SO_2$  mole fraction increases. Again, this is because higher  $SO_2$  levels result in lower oxygen levels. These lower oxygen levels result in larger flow rates simply to allow the appropriate oxygen needed to complete the reaction. Also note that the flow rates A10, S1,



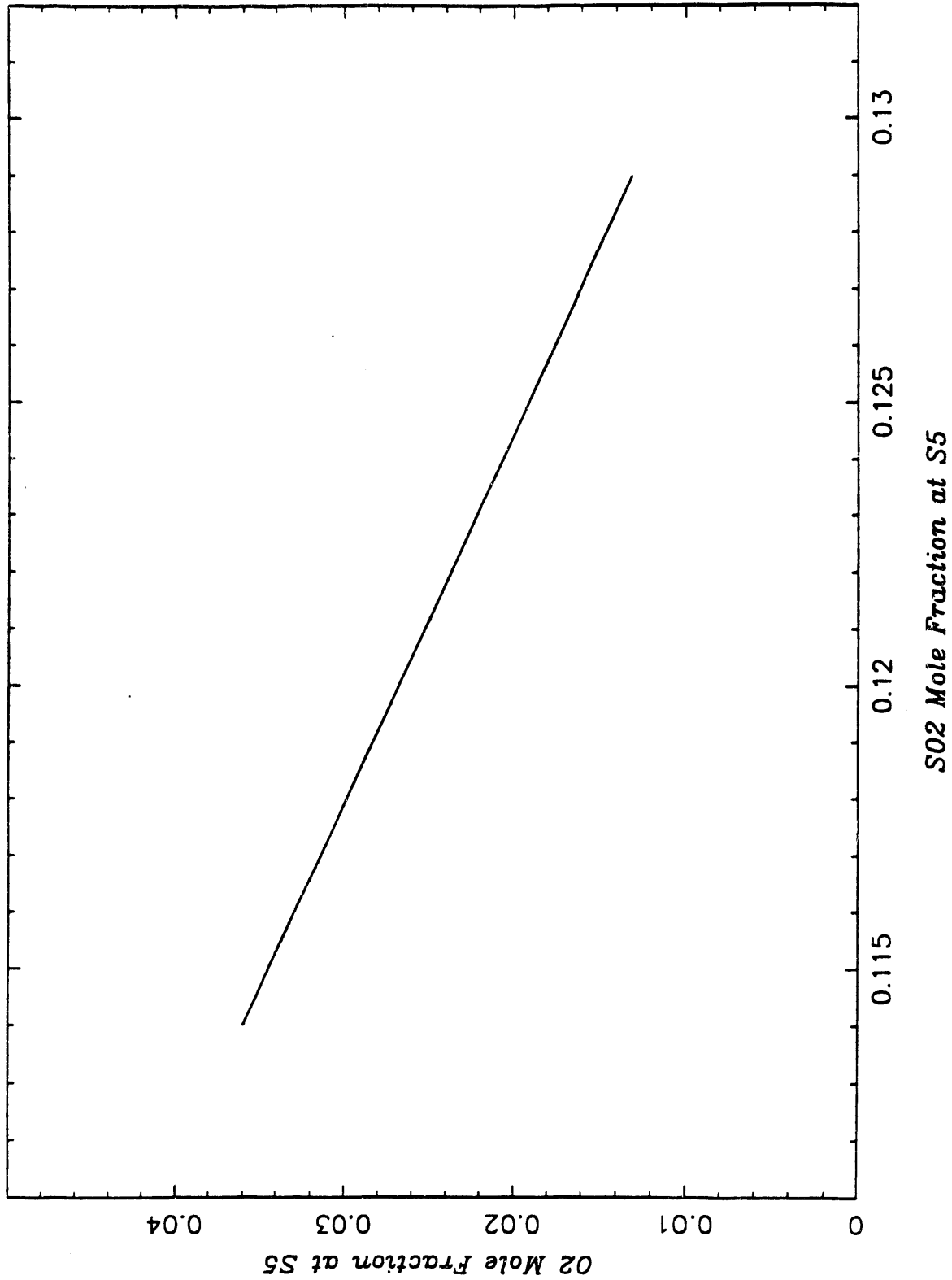


Figure 4.2 O<sub>2</sub> Mole Fraction at S5 versus SO<sub>2</sub> Mole Fraction at S5

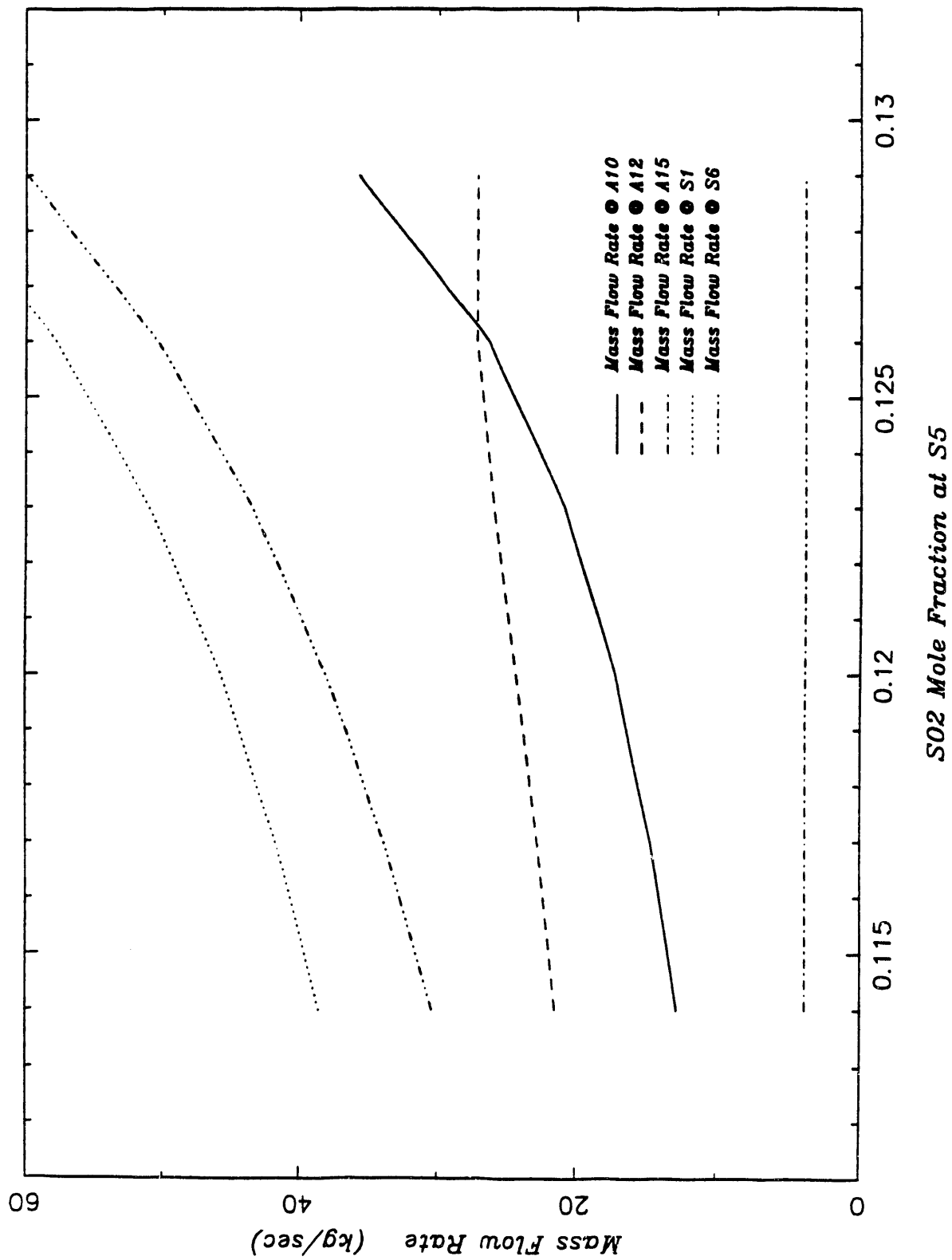


Figure 4.3 Mass Flow Rates of Regeneration Cycle versus SO<sub>2</sub> Mole Fraction at S5

and S6 increase at an increasing rate. Flow rates as high as 60 kg/sec can be very costly when considering compressor needs and therefore should be cautioned against. Mass flow rate A15 remains constant because it is used as a cooling stream and does not contribute to the oxidation reaction.

Figure 4.4 illustrates the change in effectiveness (i.e. size) of heat exchangers 1, 2, and 3 as the  $SO_2$  mole fraction at S5 is varied. Heat exchanger 1 increases in needed effectiveness as the  $SO_2$  mole fraction increases. This is mostly due to the increase in temperature of the S10-S13-S15-A10 stream needed to keep the first zone at 1250 °F. At an  $SO_2$  mole fraction of .114, the temperature needed at A10 is 654.60 °F whereas an  $SO_2$  mole fraction of .129 needs a temperature of 1036.60 °F. In the first case a lower temperature was needed to counter the effect of a higher  $O_2$  mole fraction concentration (low  $SO_2$  level). The lower needed temperatures consequently did not require the larger heat exchangers that would be needed for higher temperatures.

In the shadow of the same argument, heat exchanger 2 increases at an increasing rate as the  $SO_2$  mole fraction at S5 increases (reducing  $O_2$ ). As mentioned earlier, at lower  $SO_2$  mole fractions higher temperatures are needed in the S10 stream. These higher temperatures cause the temperature at S2 to be reduced. However, a set amount of heat must be transferred in heat exchanger 2 to increase the temperature of A7 to the intended temperature. Therefore, a lower S2 temperature will cause heat exchanger 2 to increase in size to meet the temperature needs. Under these conditions it can be seen that  $SO_2$  mole fractions higher than 12.6% may not be possible without reducing the load of heat exchanger 1 (i.e. adding heat in the S11-S12 stream).

Unlike heat exchangers 1 and 2, heat exchanger 3 slightly decreases in effectiveness as the  $SO_2$  mole fraction increases. This is due to the decrease in temperature at the S3 location by the above arguments.

Finally, as seen in Figure 4.5, a change in the  $SO_2$  mole fraction was also found to cause a change in the amount of excess oxygen at the end of zone 2. As the mole fraction of  $SO_2$  increased ( $O_2$  at S5 decreased), the amount of excess oxygen at the end of regeneration zone

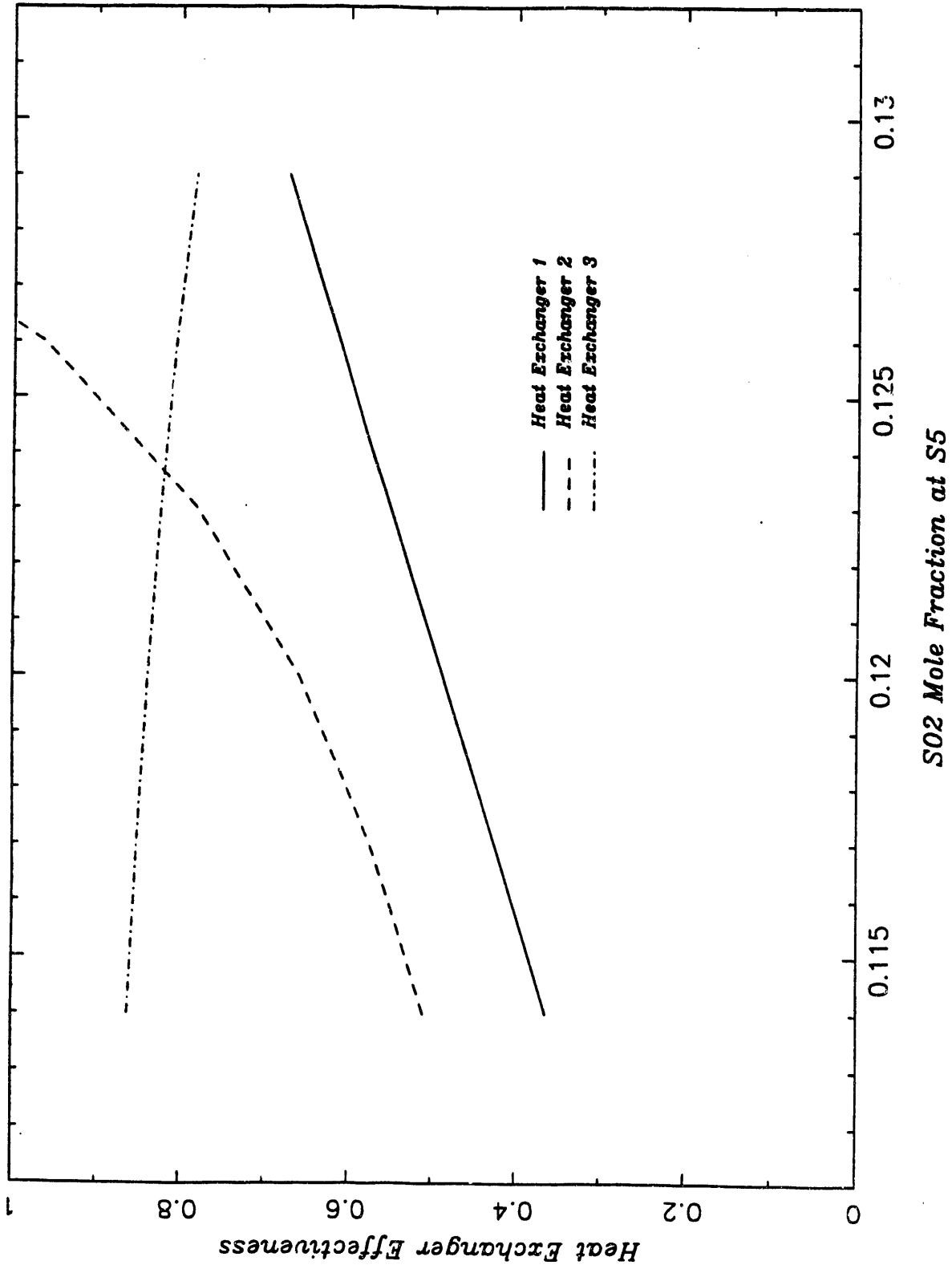


Figure 4.4 Heat Exchanger Effectiveness versus SO<sub>2</sub> Mole Fraction at S5

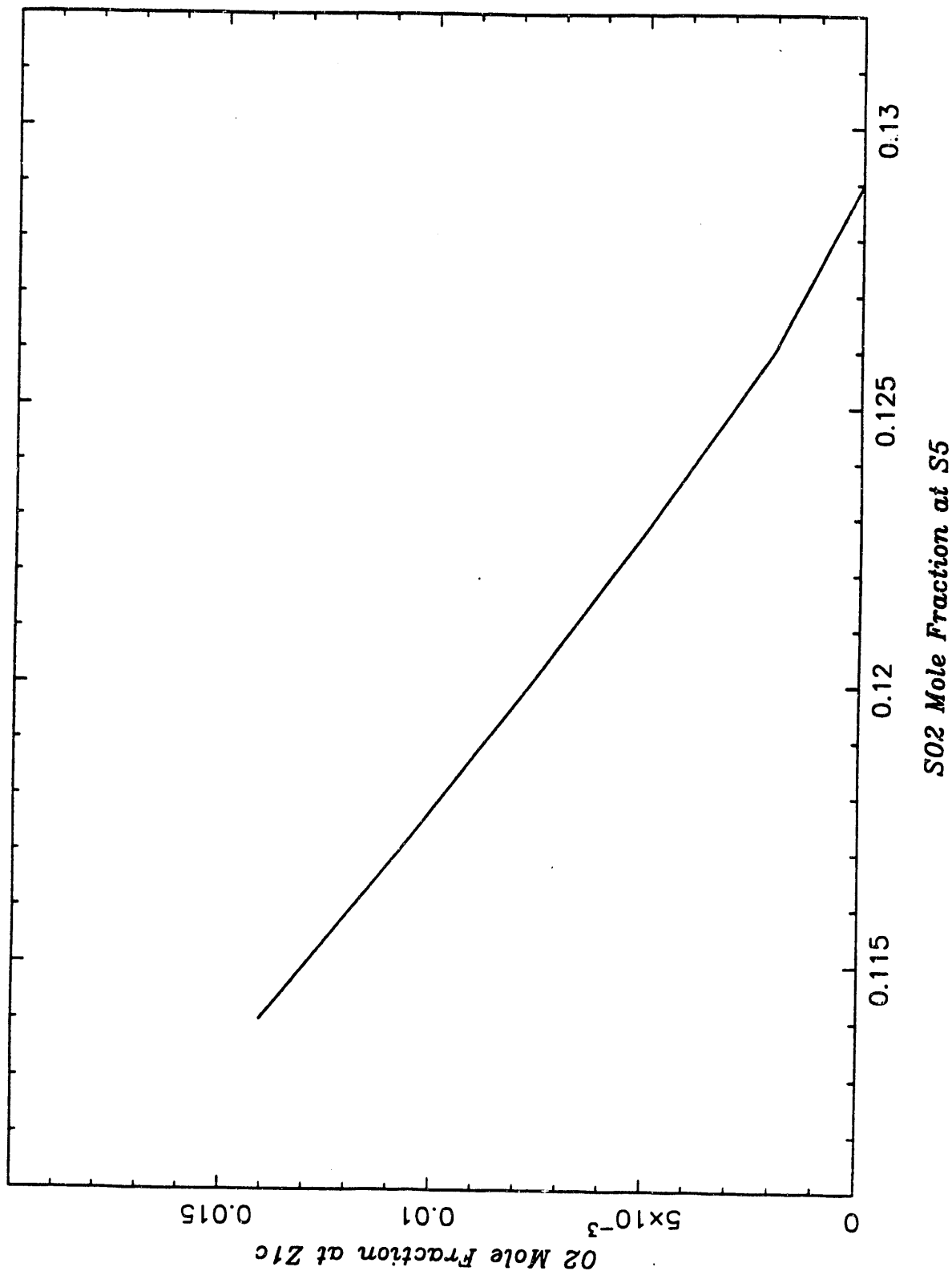
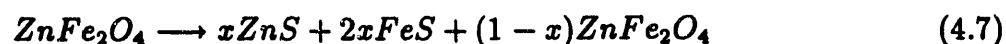


Figure 4.5 O<sub>2</sub> Mole Fraction at End of Zone 2 versus SO<sub>2</sub> Mole Fraction at S5

2 decreased. Note that the General Electric design called for about 1 percent excess oxygen at this location. Tables 4.3 through 4.8 show how variations in the  $SO_2$  concentration at S5 affect flow variables at their locations in the system.

#### 4.4.2 Percent Sorbent Reacted

A second parameter that was analyzed was the "percent of the sorbent reacted". This parameter is defined in the following equation:



where 'x' represents the percent of the sorbent reacted. If x equals zero there will be only  $ZnFe_2O_4$ . If x equals 1, there will be only  $ZnS$  and  $FeS$  entering the regeneration system. Note that this is written with respect to a sorbent with no inert binding material. The percent of the charged sorbent in Figure 4.6, 4.7, and 4.8 is related to 'x' by:

$$\%Charged\ Sorbent = x/(1 + \% Inerts) \quad (4.8)$$

Figure 4.6 illustrates that the flow rates generally decrease as the percent sorbent reacted increases. This is largely due to the fact that less flow is needed to cool the reaction to the intended 1250 °F or 1450 °F because the mass flow rate of the sorbent is less. The flow rate of the sorbent is less because there is a larger percent of reacted sulfur. The flow rate of the A10 stream was unchanged, however. This is because the rate of this flow is dictated by a mandatory 30 percent reaction in zone 1. In the computer model this is found to be a function of the amount of inlet sulfur, which is a constant.

Figure 4.7 shows the required heat exchanger effectiveness verses the percent of sorbent reacted. There does not exist the extreme changes in heat exchanger effectiveness as found in the  $SO_2$  mole fraction variations. However, it can be seen that the required effectiveness of heat exchanger 1 and 2 decrease as the percent of sorbent reacted increases. Conversely, the effectiveness of heat exchanger 3 increases as the sorbent has a larger percent of sulfur.

**Table 4.3**  
**SO2 Mole Fraction at S5**  
**(y SO2 = .114)**

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.77	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.77	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.77	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.77	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.77	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.77	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.77	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	12.80	654.60	310.00	1.0097	0.0360	0.8500	0.1140
A11	3.92	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	21.54	675.61	310.00	1.0111	0.0707	0.8381	0.0913
A13	7.77	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	38.49	1423.71	306.00	1.0097	0.0360	0.8500	0.1140
S2	38.49	1076.14	302.00	1.0097	0.0360	0.8500	0.1140
S3	38.49	1011.40	298.00	1.0097	0.0360	0.8500	0.1140
S4	38.49	200.00	294.00	1.0097	0.0360	0.8500	0.1140
S5	8.07	200.00	294.00	1.0097	0.0360	0.8500	0.1140
S6	30.42	200.00	294.00	1.0097	0.0360	0.8500	0.1140
S7	30.42	200.00	294.00	1.0097	0.0360	0.8500	0.1140
S8	30.42	214.75	314.00	1.0097	0.0360	0.8500	0.1140
S9	30.42	214.75	314.00	1.0097	0.0360	0.8500	0.1140
S10	30.42	654.60	310.00	1.0097	0.0360	0.8500	0.1140
S11	0.00	214.75	314.00	1.0097	0.0360	0.8500	0.1140
S12	0.00	214.75	310.00	1.0097	0.0360	0.8500	0.1140
S13	30.42	654.60	310.00	1.0097	0.0360	0.8500	0.1140
S14	17.62	654.60	310.00	1.0097	0.0360	0.8500	0.1140
S15	12.80	654.60	310.00	1.0097	0.0360	0.8500	0.1140
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.75	NC	"			
W3	745.00	80.13	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	17.69	1249.93	314.00	0.9313	0.0000	0.7966	0.1270
Z1B	39.23	1249.93	314.00	0.9805	0.0436	0.8222	0.1050
Z1C	39.23	1449.87	314.00	0.9911	0.0138	0.8425	0.1261
Z1D	0.74	1449.87	314.00	0.9911	0.0138	0.8425	0.1261
Z2	4.58	900.00	314.00				
Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors: 8.88E-16 S balance			
Heat Exchanger 1	N/A	0.364		-0.00000 0.00000 -0.00000			
Heat Exchanger 2	N/A	0.510		0.00000 -0.00000 0.00000			
Heat Exchanger 3	N/A	0.862		-0.00000 0.00000 -0.00000			
Heat Exchanger 4	N/A	0.000		0.00000 above error			

Table 4.4

SO2 Mole Fraction at S5  
(y SO2 = .117)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.58	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.58	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.58	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.58	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.58	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.58	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.58	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	14.71	731.00	310.00	1.0096	0.0314	0.8516	0.1170
A11	3.74	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	22.89	737.37	310.00	1.0108	0.0635	0.8406	0.0960
A13	7.58	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	41.75	1425.23	306.00	1.0096	0.0314	0.8516	0.1170
S2	41.75	1006.54	302.00	1.0096	0.0314	0.8516	0.1170
S3	41.75	948.27	298.00	1.0096	0.0314	0.8516	0.1170
S4	41.75	200.00	294.00	1.0096	0.0314	0.8516	0.1170
S5	7.89	200.00	294.00	1.0096	0.0314	0.8516	0.1170
S6	33.86	200.00	294.00	1.0096	0.0314	0.8516	0.1170
S7	33.86	200.00	294.00	1.0096	0.0314	0.8516	0.1170
S8	33.86	214.75	314.00	1.0096	0.0314	0.8516	0.1170
S9	33.86	214.75	314.00	1.0096	0.0314	0.8516	0.1170
S10	33.86	731.00	310.00	1.0096	0.0314	0.8516	0.1170
S11	0.00	214.75	314.00	1.0096	0.0314	0.8516	0.1170
S12	0.00	214.75	310.00	1.0096	0.0314	0.8516	0.1170
S13	33.86	731.00	310.00	1.0096	0.0314	0.8516	0.1170
S14	19.15	731.00	310.00	1.0096	0.0314	0.8516	0.1170
S15	14.71	731.00	310.00	1.0096	0.0314	0.8516	0.1170
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.49	NC	"			
W3	745.00	80.13	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	19.60	1249.25	314.00	0.9406	0.0000	0.8045	0.1283
Z1B	42.49	1249.25	314.00	0.9827	0.0381	0.8261	0.1089
Z1C	42.49	1449.24	314.00	0.9925	0.0106	0.8448	0.1284
Z1D	0.74	1449.24	314.00	0.9925	0.0106	0.8448	0.1284
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850		N/A		0.500		
Last Stages	0.850		N/A		Mass Flow Rate of Sulfur		
Compressor 2	0.950		N/A		7258 lb/hr		
Intercooler	N/A		0.958		Errors:		
Heat Exchanger 1	N/A		0.426		0.00E+00	S balance	
Heat Exchanger 2	N/A		0.574		-0.00000	0.00000	-0.00000
Heat Exchanger 3	N/A		0.852		0.00000	-0.00000	0.00000
Heat Exchanger 4	N/A		0.000		-0.00000	0.00000	-0.00000
					0.00000	above error	



Table 4.5

SO2 Mole Fraction at S5  
(y SO2 = .120)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	934.15	302.00	1.0094	0.0268	0.8532	0.1200
S3	43.82	882.28	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.14	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:			Percent of Complete Reaction
First Stages	0.850	N/A	0.500
Last Stages	0.850	N/A	Mass Flow Rate of Sulfur
Compressor 2	0.850	N/A	7258 lb/hr
Intercooler	N/A	0.958	Errors:
Heat Exchanger 1	N/A	0.489	0.00E+00 S balance
Heat Exchanger 2	N/A	0.660	-0.00000 0.00000 0.00000
Heat Exchanger 3	N/A	0.840	-0.00000 -0.00000 -0.00000
Heat Exchanger 4	N/A	0.000	-0.00000 0.00000 0.00000
			0.00000 above error

Table 4.6

SO2 Mole Fraction at S5  
(y SO2 = .123)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.24	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.24	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.24	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.24	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.24	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.24	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.24	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	20.89	885.00	310.00	1.0092	0.0222	0.8548	0.1230
A11	3.39	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	26.00	869.99	310.00	1.0103	0.0494	0.8454	0.1052
A13	7.24	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	51.04	1430.03	306.00	1.0092	0.0222	0.8548	0.1230
S2	51.04	858.89	302.00	1.0092	0.0222	0.8548	0.1230
S3	51.04	813.36	298.00	1.0092	0.0222	0.8548	0.1230
S4	51.04	200.00	294.00	1.0092	0.0222	0.8548	0.1230
S5	7.55	200.00	294.00	1.0092	0.0222	0.8548	0.1230
S6	43.49	200.00	294.00	1.0092	0.0222	0.8548	0.1230
S7	43.49	200.00	294.00	1.0092	0.0222	0.8548	0.1230
S8	43.49	214.75	314.00	1.0092	0.0222	0.8548	0.1230
S9	43.49	214.75	314.00	1.0052	0.0222	0.8548	0.1230
S10	43.49	885.00	310.00	1.0092	0.0222	0.8548	0.1230
S11	0.00	214.75	314.00	1.0092	0.0222	0.8548	0.1230
S12	0.00	214.75	310.00	1.0092	0.0222	0.8548	0.1230
S13	43.49	885.00	310.00	1.0092	0.0222	0.8548	0.1230
S14	22.61	885.00	310.00	1.0092	0.0222	0.8548	0.1230
S15	20.89	885.00	310.00	1.0092	0.0222	0.8548	0.1230
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.02	NC	"			
W3	745.00	80.15	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	25.78	1249.83	314.00	0.9596	0.0000	0.8207	0.1309
Z1B	51.78	1249.83	314.00	0.9874	0.0271	0.8343	0.1168
Z1C	51.78	1449.50	314.00	0.9954	0.0047	0.8495	0.1327
Z1D	0.74	1449.50	314.00	0.9954	0.0047	0.8495	0.1327
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction	
First Stages	0.850	N/A		0.500	
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur	
Compressor 2	0.850	N/A		7258 lb/hr	
Intercooler	N/A	0.958		Errors:	0.00E+00 S balance
Heat Exchanger 1	N/A	0.552			0.00000 -0.00000 0.00000
Heat Exchanger 2	N/A	0.782			-0.00000 0.00000 -0.00000
Heat Exchanger 3	N/A	0.825			0.00000 -0.00000 0.00000
Heat Exchanger 4	N/A	0.000			0.00000 above error

Table 4.7

SO2 Mole Fraction at S5  
(y SO2 = .126)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.08	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.08	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.08	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.08	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.08	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.08	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.08	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	26.38	961.10	310.00	1.0090	0.0176	0.8564	0.1260
A11	3.23	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	27.29	938.45	310.00	1.0100	0.0430	0.8476	0.1094
A13	7.08	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	57.83	1432.85	306.00	1.0090	0.0176	0.8564	0.1260
S2	57.83	781.85	302.00	1.0090	0.0176	0.8564	0.1260
S3	57.83	742.54	298.00	1.0090	0.0176	0.8564	0.1260
S4	57.83	200.00	294.00	1.0090	0.0176	0.8564	0.1260
S5	7.39	200.00	294.00	1.0090	0.0176	0.8564	0.1260
S6	50.44	200.00	294.00	1.0090	0.0176	0.8564	0.1260
S7	50.44	200.00	294.00	1.0090	0.0176	0.8564	0.1260
S8	50.44	214.75	314.00	1.0090	0.0176	0.8564	0.1260
S9	50.44	214.75	314.00	1.0090	0.0176	0.8564	0.1260
S10	50.44	961.10	310.00	1.0090	0.0176	0.8564	0.1260
S11	0.00	214.75	314.00	1.0090	0.0176	0.8564	0.1260
S12	0.00	214.75	310.00	1.0090	0.0176	0.8564	0.1260
S13	50.44	961.10	310.00	1.0090	0.0176	0.8564	0.1260
S14	24.06	961.10	310.00	1.0090	0.0176	0.8564	0.1260
S15	26.38	961.10	310.00	1.0090	0.0176	0.8564	0.1260
W1	60.00	70.00	NC	No Gas			
W2	60.00	79.79	NC	"			
W3	745.00	80.17	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	31.27	1249.92	314.00	0.9693	0.0000	0.8291	0.1322
Z1B	58.56	1249.92	314.00	0.9898	0.0217	0.8384	0.1207
Z1C	58.56	1449.95	314.00	0.9969	0.0020	0.8519	0.1347
Z1D	0.74	1449.95	314.00	0.9969	0.0020	0.8519	0.1347
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	8.88E-16	S balance	
Heat Exchanger 1	N/A	0.613		0.00000	0.00000	-0.00000	
Heat Exchanger 2	N/A	0.964		-0.00000	-0.00000	0.00000	
Heat Exchanger 3	N/A	0.807		0.00000	0.00000	-0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.8

**SO<sub>2</sub> Mole Fraction at 85  
(y SO<sub>2</sub> = .129)**

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O <sub>2</sub>	N <sub>2</sub>	SO <sub>2</sub>
A1	6.93	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	6.93	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	6.93	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	6.93	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	6.93	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	6.93	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	6.93	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	35.73	1036.60	310.00	1.0089	0.0131	0.8579	0.1290
A11	3.08	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	27.25	1006.44	310.00	1.0098	0.0379	0.8494	0.1127
A13	6.93	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	67.14	1435.30	306.00	1.0089	0.0131	0.8579	0.1290
S2	67.14	702.03	302.00	1.0089	0.0131	0.8579	0.1290
S3	67.14	668.89	298.00	1.0089	0.0131	0.8579	0.1290
S4	67.14	200.00	294.00	1.0089	0.0131	0.8579	0.1290
S5	7.24	200.00	294.00	1.0089	0.0131	0.8579	0.1290
S6	59.90	200.00	294.00	1.0089	0.0131	0.8579	0.1290
S7	59.90	200.00	294.00	1.0089	0.0131	0.8579	0.1290
S8	59.90	214.75	314.00	1.0089	0.0131	0.8579	0.1290
S9	59.90	214.75	314.00	1.0089	0.0131	0.8579	0.1290
S10	59.90	1036.60	310.00	1.0089	0.0131	0.8579	0.1290
S11	0.00	214.75	314.00	1.0089	0.0131	0.8579	0.1290
S12	0.00	214.75	310.00	1.0089	0.0131	0.8579	0.1290
S13	59.90	1036.60	310.00	1.0089	0.0131	0.8579	0.1290
S14	24.17	1036.60	310.00	1.0089	0.0131	0.8579	0.1290
S15	35.73	1036.60	310.00	1.0089	0.0131	0.8579	0.1290
W1	60.00	70.00	NC	No Gas			
W2	60.00	79.58	NC	"			
W3	745.00	80.20	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	40.62	1249.85	314.00	0.9792	0.0000	0.8375	0.1336
Z1B	67.87	1249.85	314.00	0.9924	0.0163	0.8426	0.1246
Z1C	67.87	1449.95	314.00	0.9984	-0.0006	0.8542	0.1366
Z1D	0.74	1449.95	314.00	0.9984	-0.0006	0.8542	0.1366
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850	N/A			0.500		
Last Stages	0.850	N/A			Mass Flow Rate of Sulfur		
Compressor 2	0.850	N/A			7258 lb/hr		
Intercooler	N/A	0.958			Errors:		
Heat Exchanger 1	N/A	0.673			0.00E+00	S balance	
Heat Exchanger 2	N/A	1.271			-0.00000	0.00000	-0.00000
Heat Exchanger 3	N/A	0.783			0.00000	-0.00000	0.00000
Heat Exchanger 4	N/A	0.000			-0.00000	0.00000	-0.00000
					0.00000	above error	

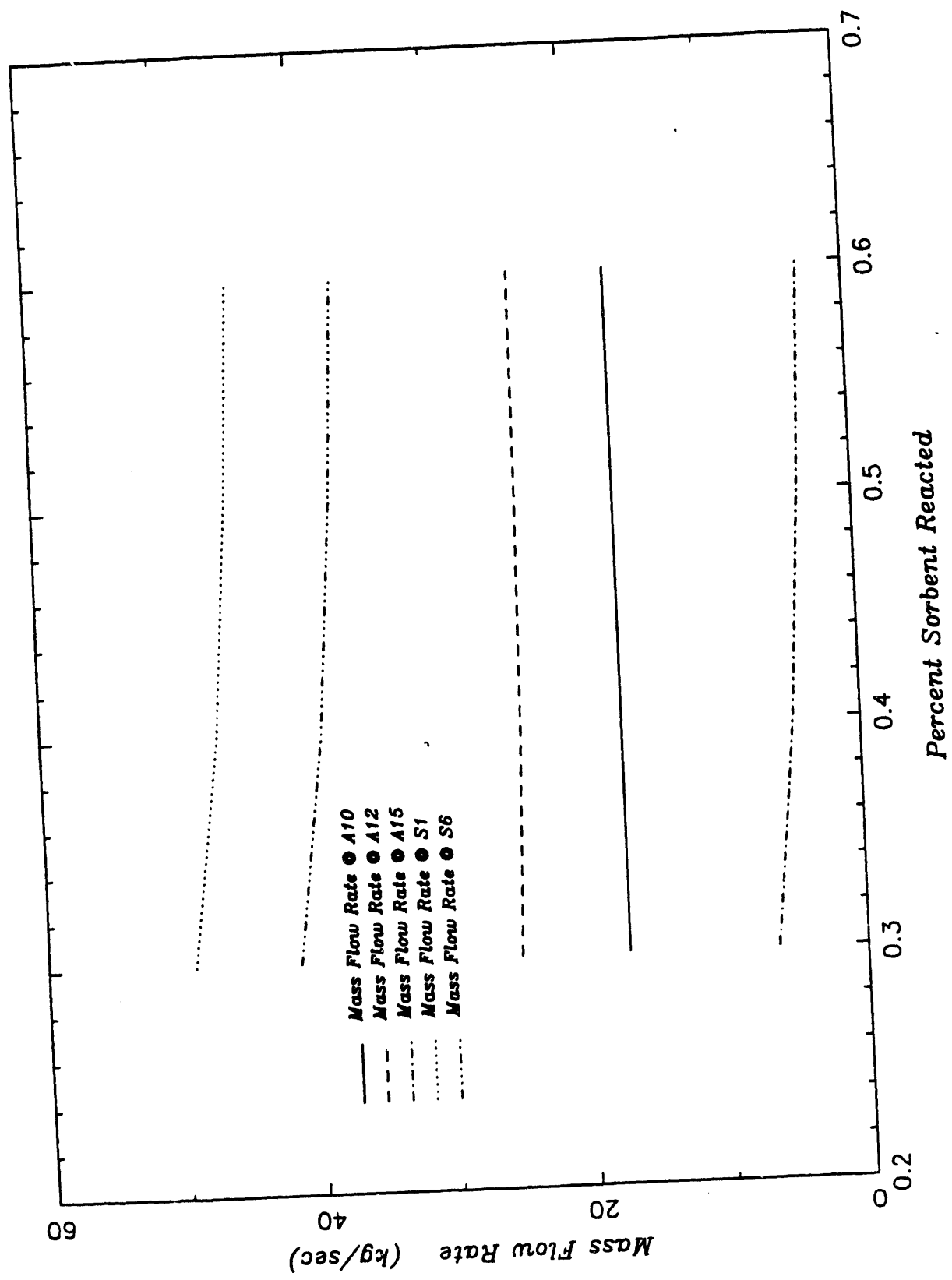


Figure 4.6 Mass Flow Rates of Regeneration Cycle Verses Percent of Inlet Sorbent Reacted

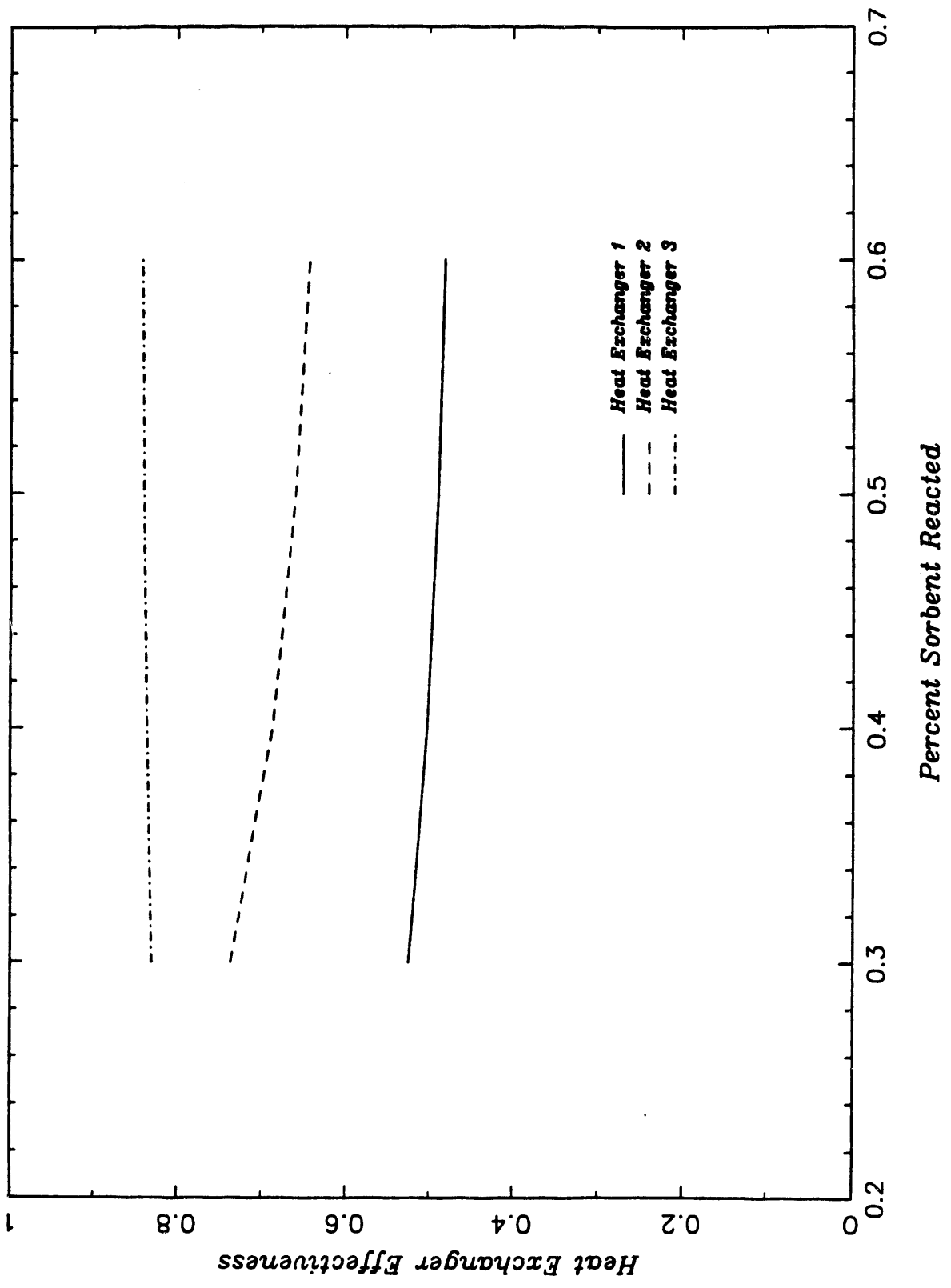


Figure 4.7 Heat Exchanger Effectiveness Verses Percent of Inlet Sorbent Reacted

The excess amount of oxygen at the end of zone 2 versus the percent of sorbent reacted can be seen in Figure 4.8. In this illustration the excess amount of oxygen increases at a decreasing rate as the percent sorbent reacted increases. Note that the excess amount of oxygen falls below zero when the percent sorbent reacted is around 0.36. This should not be taken as a limit due to the fact that many other parameters can also affect the percentage of excess oxygen. The purpose of this illustration is to analyze the results of a parameter change while all other independent variables are held constant. By changing other parameters, scenarios may be obtained that show excess oxygen at less than 0.30 percent sorbent reacted. Tables 4.9 through 4.12 show how the system flow variables depend on the percent of sorbent reacted.

#### 4.4.3 Temperature at A7

The temperature at A7 is also a parameter that can influence the heat exchanger effectiveness and mass flow rates through the regeneration system. In Figure 4.9 it can be seen that the heat exchanger effectiveness at heat exchanger 2 increases greatly as the temperature at A7 is increased. However, heat exchangers 1 and 3 remain generally unchanged.

Figure 4.10 illustrates the changes in mass flow rates of important streams. It can be seen that the flow of stream A10 remains constant. Once again, this is due to the fact that the flow rate of this stream is dictated by oxygen requirements. The mass flow rates of streams A12 and A15 are slightly increased as the temperature of A7 increases. This is due to a more difficult job of cooling the reaction zones to 1250 °F and 1450 °F. The higher the inlet temperature, the larger the flow that is needed for cooling purposes. Finally, higher flow rates at streams S1 and S6 are a consequence of the higher flow rates needed earlier in the system. Tables 4.13 through 4.18 provide additional detailed information.

Table 4.9

Percent Sorbent Reacted  
30 %

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	844.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	1.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	25.13	841.06	310.00	1.0097	0.0349	0.8504	0.1147
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	6.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	6.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	49.12	1414.23	306.00	1.0094	0.0268	0.8532	0.1200
S2	49.12	883.81	302.00	1.0094	0.0268	0.8532	0.1200
S3	49.12	835.42	298.00	1.0094	0.0268	0.8532	0.1200
S4	49.12	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	41.40	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	41.40	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	41.40	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	41.40	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	41.40	844.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	41.40	844.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	24.13	844.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	844.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.12	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	7.95	1000.00	310.00	"			
Z1A	25.22	1249.99	314.00	0.9290	0.0000	0.7945	0.1267
Z1B	50.35	1249.99	314.00	0.9755	0.0201	0.8267	0.1198
Z1C	50.35	1449.85	314.00	0.9840	-0.0046	0.8434	0.1373
Z1D	1.23	1449.85	314.00	0.9840	-0.0046	0.8434	0.1373
Z2	7.64	900.00	314.00				

Compressor 1:  
 First Stages 0.850  
 Last Stages 0.850  
 Compressor 2 0.850  
 Intercooler N/A  
 Heat Exchanger 1 N/A  
 Heat Exchanger 2 N/A  
 Heat Exchanger 3 N/A  
 Heat Exchanger 4 N/A

N/A  
 N/A  
 N/A  
 0.958  
 0.525  
 0.737  
 0.830  
 0.000

Percent of Complete Reaction  
0.300

Mass Flow Rate of Sulfur  
7258 lb/hr

Errors: 0.00E+00 S balance  
 0.00000 -0.00000 -0.00000  
 -0.00000 0.00000 0.00000  
 0.00000 -0.00000 -0.00000  
 0.00000 above error



Table 4.10

**Percent Sorbent Reacted**  
40 %

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	822.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	2.60	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.63	816.51	310.00	1.0102	0.0483	0.8458	0.1059
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	4.81	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	4.81	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	47.01	1422.87	306.00	1.0094	0.0268	0.8532	0.1200
S2	47.01	915.25	302.00	1.0094	0.0268	0.8532	0.1200
S3	47.01	864.70	298.00	1.0094	0.0268	0.8532	0.1200
S4	47.01	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	39.30	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	39.30	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	39.30	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	39.30	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	39.30	822.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	39.30	822.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	22.03	822.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	822.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.13	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	6.04	1000.00	310.00	"			
Z1A	23.31	1250.00	314.00	0.9420	0.0000	0.8057	0.1285
Z1B	47.94	1250.00	314.00	0.9814	0.0279	0.8288	0.1155
Z1C	47.94	1449.94	314.00	0.9901	0.0029	0.8457	0.1331
Z1D	0.92	1449.94	314.00	0.9901	0.0029	0.8457	0.1331
Z2	5.73	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.400			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:			
Heat Exchanger 1	N/A	0.503		0.00000	-0.00000	0.00000	S balance
Heat Exchanger 2	N/A	0.687		0.00000	0.00000	-0.00000	
Heat Exchanger 3	N/A	0.836		-0.00000	-0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.11

Percent Sorbent Reacted  
50 %

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	934.15	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	882.28	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.14	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:			
First Stages	0.850	N/A	
Last Stages	0.850	N/A	
Compressor 2	0.850	N/A	
Intercooler	N/A	0.958	
Heat Exchanger 1	N/A	0.489	
Heat Exchanger 2	N/A	0.660	
Heat Exchanger 3	N/A	0.840	
Heat Exchanger 4	N/A	0.000	

Percent of Complete Reaction  
0.500  
Mass Flow Rate of Sulfur  
7258 lb/hr

Errors: 0.00E+00 S balance  
-0.00000 0.00000 0.00000  
0.00000 -0.00000 -0.00000  
-0.00000 0.00000 0.00000  
0.00000 above error

Table 4.12

Percent Sorbent Reacted  
60 %

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	799.90	310.00	1.0094	0.0268	0.8532	0.1200
A11	4.20	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.27	794.72	310.00	1.0107	0.0617	0.8412	0.0971
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.21	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.21	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.05	1431.76	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.05	946.80	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.05	894.04	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.05	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	37.34	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	37.34	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	37.34	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	37.34	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	37.34	799.90	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	37.34	799.90	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.07	799.90	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	799.90	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.14	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.13	1000.00	310.00	"			
Z1A	21.40	1249.93	314.00	0.9554	0.0000	0.8172	0.1303
Z1B	45.67	1249.93	314.00	0.9875	0.0357	0.8311	0.1111
Z1C	45.67	1449.98	314.00	0.9964	0.0107	0.8482	0.1288
Z1D	0.61	1449.98	314.00	0.9964	0.0107	0.8482	0.1288
Z2	3.82	900.00	314.00				

Compressor 1:			
First Stages	0.850	N/A	
Last Stages	0.850	N/A	
Compressor 2	0.850	N/A	
Intercooler	N/A	0.958	
Heat Exchanger 1	N/A	0.481	
Heat Exchanger 2	N/A	0.643	
Heat Exchanger 3	N/A	0.842	
Heat Exchanger 4	N/A	0.000	

Percent of Complete Reaction  
0.600  
Mass Flow Rate of Sulfur  
7258 lb/hr

Errors: 0.00E+00 S balance  
-0.00000 -0.00000 0.00000  
0.00000 0.00000 -0.00000  
-0.00000 -0.00000 0.00000  
0.00000 above error

Table 4.13

Temperature at A7  
(T<sub>A7</sub>=770 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	934.15	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	882.28	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.14	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00				
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	0.00E+00	S balance	
Heat Exchanger 1	N/A	0.489		-0.00000	0.00000	0.00000	
Heat Exchanger 2	N/A	0.660		0.00000	-0.00000	-0.00000	
Heat Exchanger 3	N/A	0.840		-0.00000	0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.14

Temperature at A7  
(T<sub>A7</sub>=800 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	800.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	800.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	800.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.70	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.33	800.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.60	807.52	310.00	1.0104	0.0542	0.8437	0.1021
A13	7.41	800.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	4.08	800.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	4.08	800.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	46.26	1427.72	306.00	1.0094	0.0268	0.8532	0.1200
S2	46.26	932.82	302.00	1.0094	0.0268	0.8532	0.1200
S3	46.26	876.60	298.00	1.0094	0.0268	0.8532	0.1200
S4	46.26	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.54	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.54	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.54	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.54	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.54	808.70	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.54	808.70	310.00	1.0094	0.0268	0.8532	0.1200
S14	21.27	808.70	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.70	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.15	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.93	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.76	1249.93	314.00	0.9851	0.0314	0.8306	0.1136
Z1C	46.76	1449.47	314.00	0.9939	0.0065	0.8476	0.1313
Z1D	0.50	1449.47	314.00	0.9939	0.0065	0.8476	0.1313
Z2	4.58	900.00	314.00				

Compressor 1:  
 First Stages 0.850 N/A  
 Last Stages 0.850 N/A  
 Compressor 2 0.850 N/A  
 Intercooler N/A 0.958  
 Heat Exchanger 1 N/A 0.490  
 Heat Exchanger 2 N/A 0.724  
 Heat Exchanger 3 N/A 0.839  
 Heat Exchanger 4 N/A 0.000

Percent of Complete Reaction  
 0.500  
 Mass Flow Rate of Sulfur  
 7258 lb/hr  
 Errors: 0.00E+00 S balance  
 -0.00000 0.00000 -0.00000  
 0.00000 -0.00000 0.00000  
 -0.00000 0.00000 -0.00000  
 0.00000 above error

Table 4.15

Temperature at A7  
(T<sub>A7</sub>=830 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	830.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	830.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	830.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.76	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.06	830.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.72	811.39	310.00	1.0104	0.0519	0.8445	0.1035
A13	7.41	830.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	4.35	830.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	4.35	830.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	46.64	1428.11	306.00	1.0094	0.0268	0.8532	0.1200
S2	46.64	932.35	302.00	1.0094	0.0268	0.8532	0.1200
S3	46.64	871.79	298.00	1.0094	0.0268	0.8532	0.1200
S4	46.64	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.93	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.93	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.93	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.93	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.93	808.76	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.93	808.76	310.00	1.0094	0.0268	0.8532	0.1200
S14	21.66	808.76	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.76	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.16	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.98	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.88	1249.98	314.00	0.9851	0.0302	0.8311	0.1145
Z1C	46.88	1449.90	314.00	0.9939	0.0053	0.8481	0.1321
Z1D	0.24	1449.90	314.00	0.9939	0.0053	0.8481	0.1321
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	0.00E+00	S balance	
Heat Exchanger 1	N/A	0.490		0.00000	0.00000	-0.00000	
Heat Exchanger 2	N/A	0.787		-0.00000	-0.00000	0.00000	
Heat Exchanger 3	N/A	0.838		0.00000	0.00000	-0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.16

Temperature at A7  
(T<sub>A7</sub>=860 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	860.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	860.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	860.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.78	310.00	1.0094	0.0268	0.8532	0.1200
A11	2.76	860.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.84	814.47	310.00	1.0103	0.0494	0.8454	0.1052
A13	7.41	860.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	4.65	860.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	4.65	860.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	47.06	1428.08	306.00	1.0094	0.0268	0.8532	0.1200
S2	47.06	931.42	302.00	1.0094	0.0268	0.8532	0.1200
S3	47.06	866.65	298.00	1.0094	0.0268	0.8532	0.1200
S4	47.06	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	39.35	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	39.35	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	39.35	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	39.35	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	39.35	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	39.35	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S14	22.08	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.78	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.17	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.99	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	47.00	1249.99	314.00	0.9850	0.0287	0.8317	0.1154
Z1C	47.00	1449.91	314.00	0.9939	0.0039	0.8486	0.1330
Z1D	-0.06	1449.91	314.00	0.9939	0.0039	0.8486	0.1330
Z2	4.58	900.00	314.00	"			
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors: 0.00E+00 S balance			
Heat Exchanger 1	N/A	0.490		0.00000	0.00000	-0.00000	
Heat Exchanger 2	N/A	0.851		-0.00000	-0.00000	0.00000	
Heat Exchanger 3	N/A	0.837		0.00000	0.00000	-0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.17

Temperature at A7  
(T<sub>A7</sub>=890 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	890.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	890.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	890.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.78	310.00	1.0094	0.0268	0.8532	0.1200
A11	2.41	890.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.93	816.64	310.00	1.0101	0.0465	0.8464	0.1071
A13	7.41	890.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	4.99	890.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	4.99	890.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	47.50	1427.95	306.00	1.0094	0.0268	0.8532	0.1200
S2	47.50	930.39	302.00	1.0094	0.0268	0.8532	0.1200
S3	47.50	861.51	298.00	1.0094	0.0268	0.8532	0.1200
S4	47.50	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	39.79	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	39.79	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	39.79	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	39.79	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	39.79	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	39.79	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S14	22.52	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.78	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.19	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.99	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	47.09	1249.99	314.00	0.9850	0.0271	0.8322	0.1165
Z1C	47.09	1449.84	314.00	0.9938	0.0022	0.8492	0.1341
Z1D	-0.41	1449.84	314.00	0.9938	0.0022	0.8492	0.1341
Z2	4.58	900.00	314.00				
Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors: 0.00E+00 S balance			
Heat Exchanger 1	N/A	0.490		-0.00000	-0.00000	0.00000	
Heat Exchanger 2	N/A	0.916		0.00000	0.00000	-0.00000	
Heat Exchanger 3	N/A	0.836		-0.00000	-0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		



Table 4.18

Temperature at A7  
(T<sub>A7</sub>=920 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	S02
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	920.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	920.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	920.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.78	310.00	1.0094	0.0268	0.8532	0.1200
A11	2.01	920.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.97	817.74	310.00	1.0100	0.0432	0.8475	0.1092
A13	7.41	920.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	5.40	920.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	5.40	920.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	47.94	1427.95	306.00	1.0094	0.0268	0.8532	0.1200
S2	47.94	929.51	302.00	1.0094	0.0268	0.8532	0.1200
S3	47.94	856.58	298.00	1.0094	0.0268	0.8532	0.1200
S4	47.94	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	40.22	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	40.22	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	40.22	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	40.22	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	40.22	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	40.22	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S14	22.95	808.78	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.78	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.21	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.99	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	47.13	1249.99	314.00	0.9849	0.0251	0.8329	0.1178
Z1C	47.13	1449.95	314.00	0.9937	0.0002	0.8498	0.1354
Z1D	-0.81	1449.95	314.00	0.9937	0.0002	0.8498	0.1354
Z2	4.58	900.00	314.00				

Compressor 1:

First Stages	0.850	N/A
Last Stages	0.850	N/A
Compressor 2	0.850	N/A
Intercooler	N/A	0.958
Heat Exchanger 1	N/A	0.490
Heat Exchanger 2	N/A	0.980
Heat Exchanger 3	N/A	0.835
Heat Exchanger 4	N/A	0.000

Percent of Complete Reaction  
0.500

Mass Flow Rate of Sulfur  
7258 lb/hr

Errors: 0.00E+00 S balance  
0.00000 0.00000 -0.00000  
-0.00000 -0.00000 0.00000  
0.00000 0.00000 -0.00000  
0.00000 above error

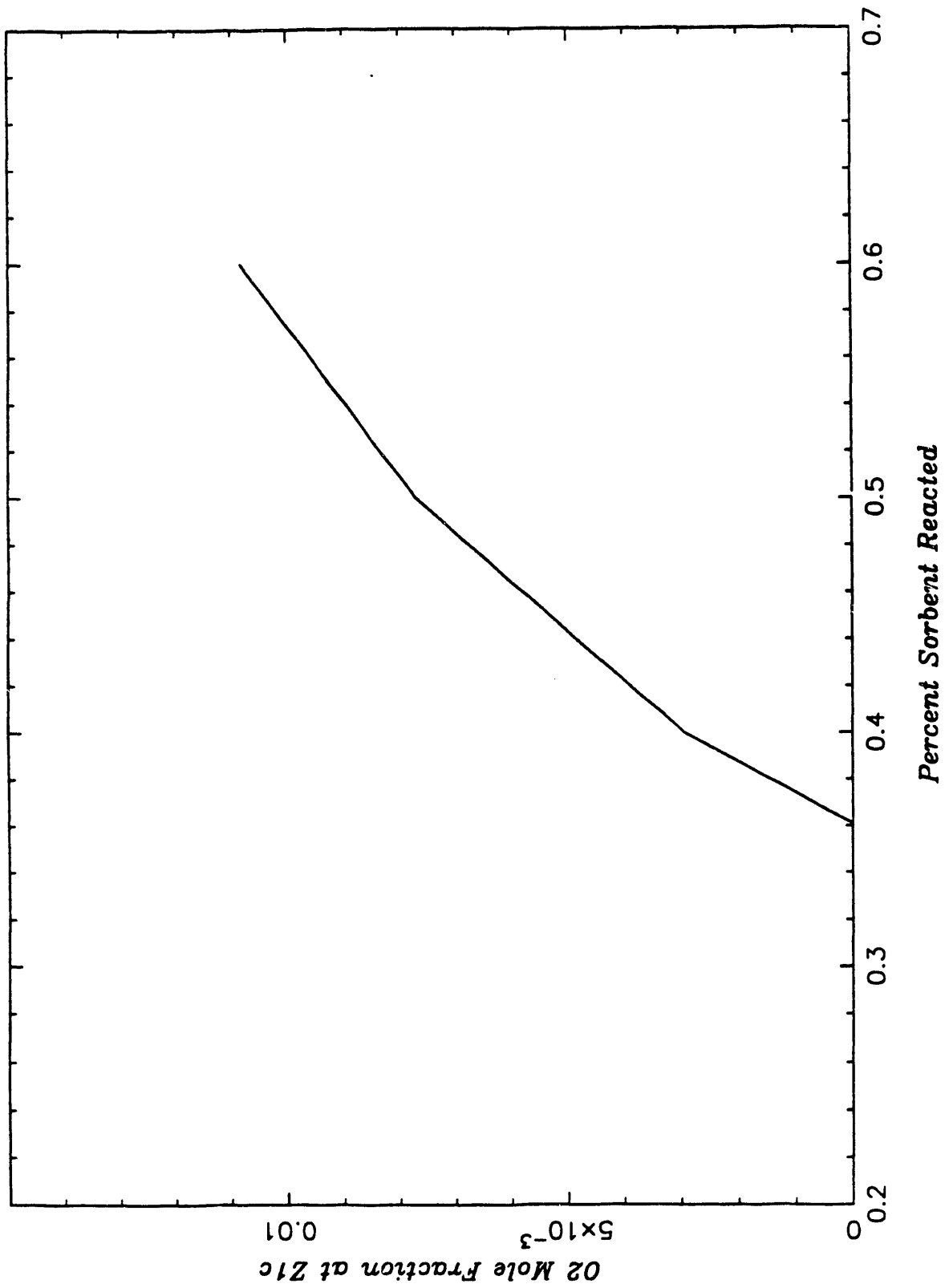


Figure 4.8 O<sub>2</sub> Mole Fraction at End of Zone 2 versus Percent of Inlet Sorbent Reacted

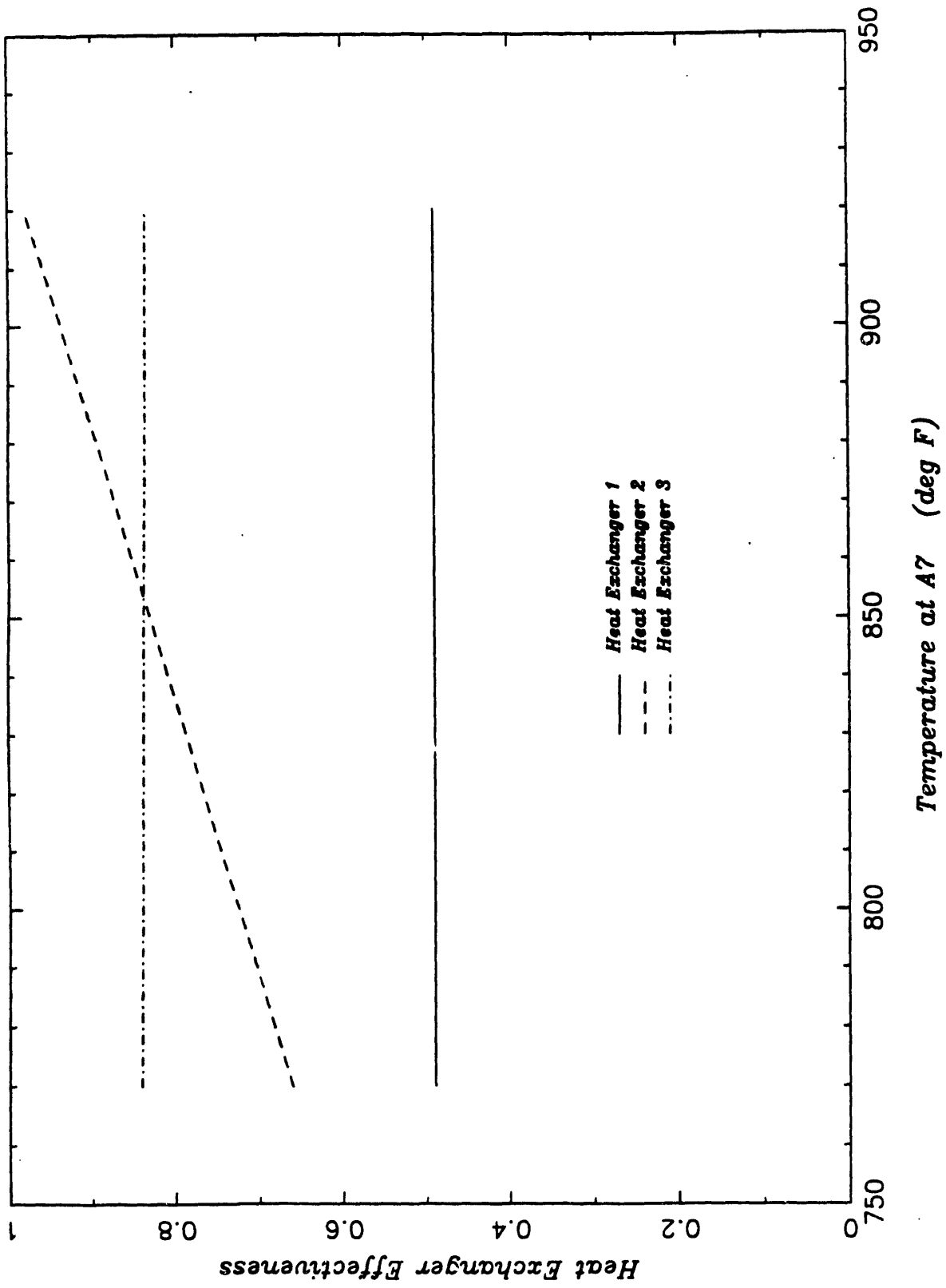


Figure 4.9 Heat Exchanger Effectiveness verses Temperature at A7

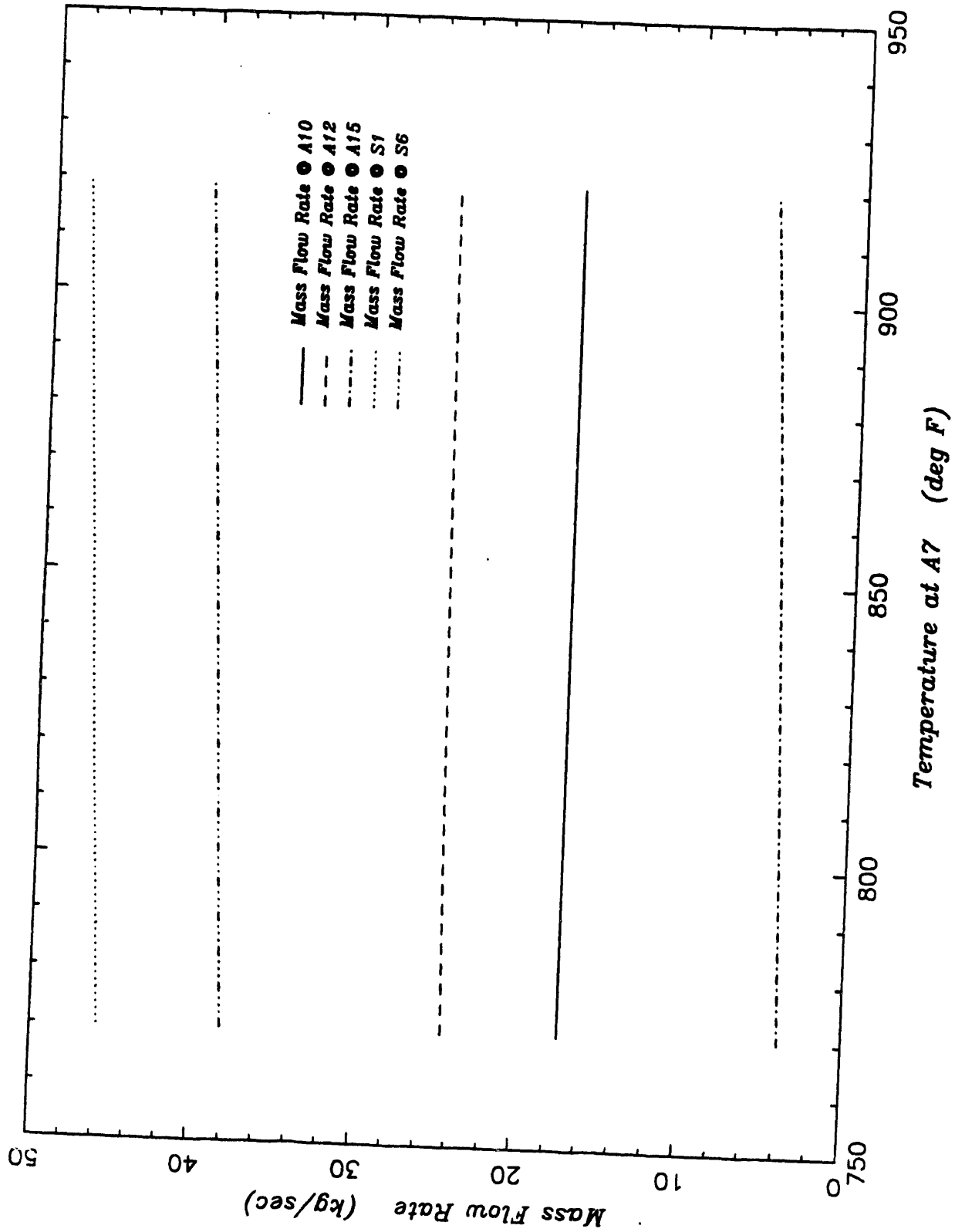


Figure 4.10 Mass Flow Rates of Regeneration Cycle versus Temperature at A7

#### 4.4.4 Temperature at S5

Figure 4.11 illustrates the needed heat exchanger effectiveness as the temperature of the S5 stream is allowed to increase. As will be seen later, it is desirable to maintain higher temperatures at the S5 bleed stream. However, as can be seen in Figure 4.11, this is highly desirable with respect to heat exchangers. Heat exchangers 1, 2, and 3 decrease in needed effectiveness (i.e. size) as the temperature of S5 increases. Note that heat exchanger 1 decreases at an increasing rate and that at S5 temperatures higher than 780 °F a heat exchanger may not even be necessary. Note also that at this temperature heat exchangers 2 and 3 have effectiveness of only 32.6% and 45.6% respectively. Tables 4.19 through 4.31 provide additional information.

#### 4.5 Direct Sulfur Removal

Research Triangle Institute (RTI) has been working on direct sulfur recovery under contract with the Department of Energy at Morgantown Energy Technology Center. Their work centers around tail-gasses produced by the regeneration of metal oxide sorbents (ie. Zinc Ferrite). Figure is a schematic of the system. Although experimental tests were performed on gas streams with  $SO_2$  concentrations around 1%-3%, not 10-13%, there appear to be only benefits when considering the more concentrated streams. In addition, RTI found that the yield of sulfur increased as temperatures were increased or when pressures increased [22]. Both of these facts are extremely beneficial to the General Electric design under consideration. Below are a few results obtained from the RTI experiments:

Table 4.32

At Constant Temperature (650 C)		
Pressure		Percentage Sulfur Removal
1 atm	( 14.7 psi)	20 %
10 atm	(147.0 psi)	60 %
20 atm	(294.0 psi)	90 %
40 atm	(588.0 psi)	95 %

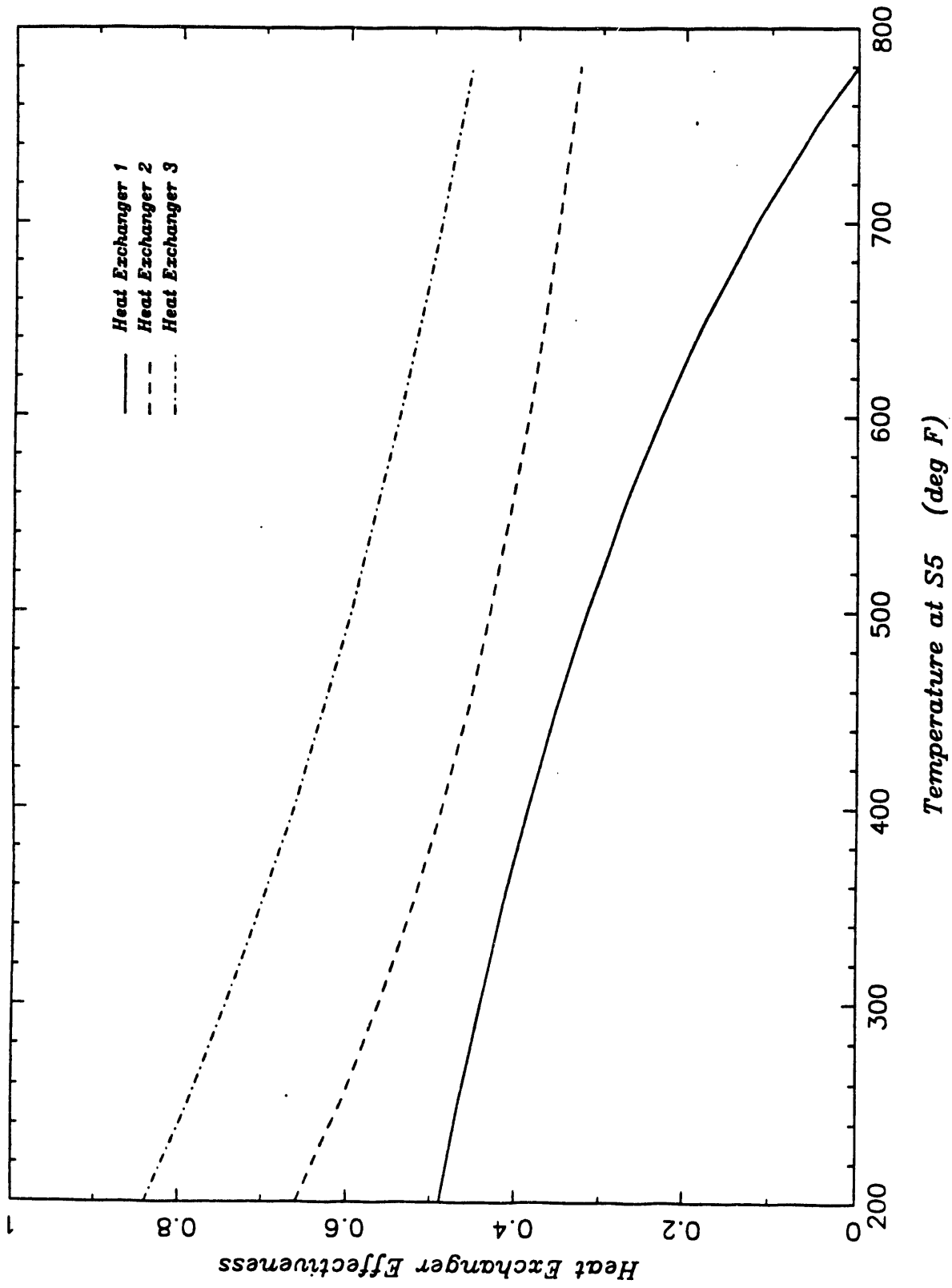


Figure 4.11 Heat Exchanger Effectiveness versus Temperature at S5

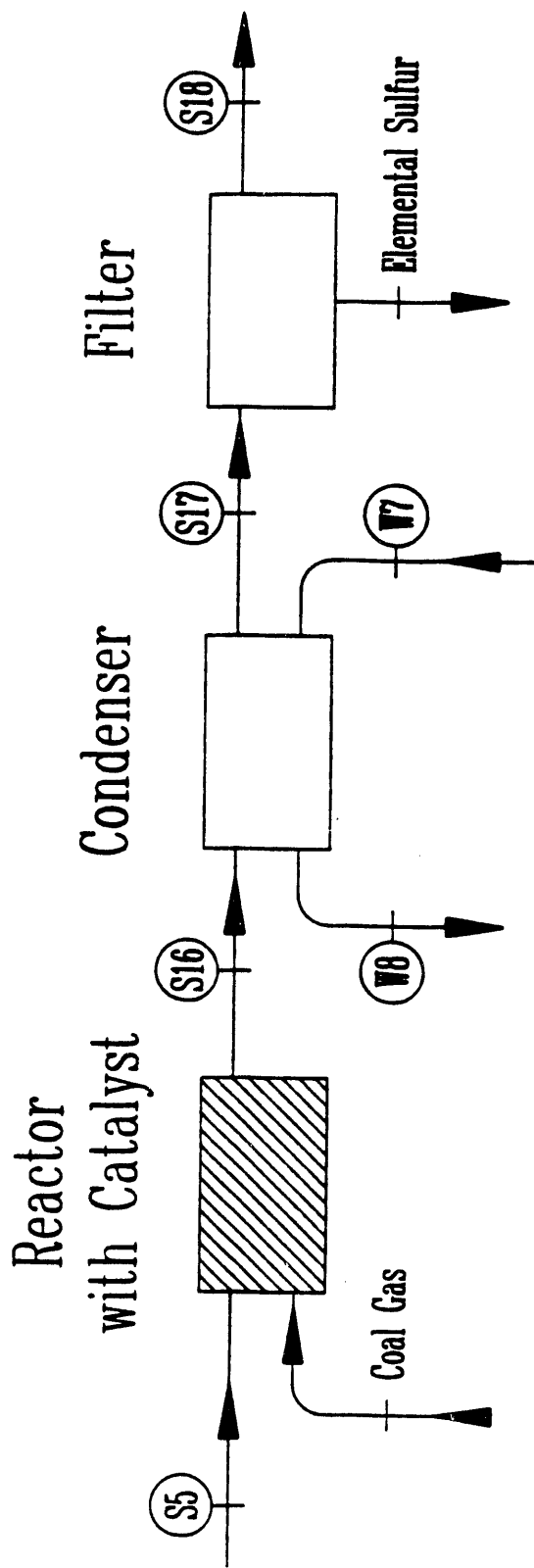


Figure 4.12 RTI Direct Surfur Recovery Loop

Table 4.19

Temperature at S5  
(T<sub>S</sub>=200 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	934.15	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	882.28	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	200.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	214.75	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	214.75	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.14	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:  
 First Stages 0.850 N/A  
 Last Stages 0.850 N/A  
 Compressor 2 0.850 N/A  
 Intercooler N/A 0.958  
 Heat Exchanger 1 N/A 0.489  
 Heat Exchanger 2 N/A 0.660  
 Heat Exchanger 3 N/A 0.840  
 Heat Exchanger 4 N/A 0.000

Percent of Complete Reaction  
 0.500

Mass Flow Rate of Sulfur  
 7258 lb/hr

Errors: 0.00E+00 S balance  
 -0.00000 0.00000 0.00000  
 0.00000 -0.00000 -0.00000  
 -0.00000 0.00000 0.00000  
 0.00000 above error



Table 4.20

**Temperature at S5**  
( $T_{S5}=250$  °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	976.67	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	924.79	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	250.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	250.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	250.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	250.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	265.87	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	265.87	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	265.87	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	265.87	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	80.03	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:			
Heat Exchanger 1	N/A	0.467		0.00E+00	S balance		
Heat Exchanger 2	N/A	0.607		-0.00000	0.00000	0.00000	
Heat Exchanger 3	N/A	0.789		0.00000	-0.00000	-0.00000	
Heat Exchanger 4	N/A	0.000		-0.00000	0.00000	0.00000	
				0.00000	above error		

Table 4.21

**Temperature at S5**  
( $T_{S5}=300$  °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1019.18	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	967.31	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	300.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	300.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	300.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	300.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	316.99	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	316.99	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	316.99	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	316.99	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.92	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction		
First Stages	0.850	N/A		0.500		
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur		
Compressor 2	0.850	N/A		7258 lb/hr		
Intercooler	N/A	0.958		Errors:		
Heat Exchanger 1	N/A	0.442		0.00000	0.00000	0.00000
Heat Exchanger 2	N/A	0.561		0.00000	-0.00000	-0.00000
Heat Exchanger 3	N/A	0.744		-0.00000	0.00000	0.00000
Heat Exchanger 4	N/A	0.000		0.00000	above error	

Table 4.22

Temperature at S5  
(T<sub>S5</sub>=350 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1061.69	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1009.82	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	350.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	350.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	350.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	350.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	368.11	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	368.11	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	368.11	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	368.11	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.80	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:  
 First Stages  
 Last Stages  
 Compressor 2  
 Intercooler  
 Heat Exchanger 1  
 Heat Exchanger 2  
 Heat Exchanger 3  
 Heat Exchanger 4

0.850 N/A  
 0.850 N/A  
 0.850 N/A  
 N/A 0.958  
 N/A 0.415  
 N/A 0.522  
 N/A 0.702  
 N/A 0.000

Percent of Complete Reaction  
 0.500  
 Mass Flow Rate of Sulfur  
 7258 lb/hr

Errors: 0.00E+00 S balance  
 -0.00000 0.00000 0.00000  
 0.00000 -0.00000 -0.00000  
 -0.00000 0.00000 0.00000  
 0.00000 above error

Table 4.23

Temperature at S5  
(T<sub>S5</sub>=400 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1104.20	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1052.33	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	400.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	400.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	400.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	400.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	419.22	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	419.22	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	419.22	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	419.22	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.69	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850		N/A		0.500		
Last Stages	0.850		N/A		Mass Flow Rate of Sulfur		
Compressor 2	0.850		N/A		7258 lb/hr		
Intercooler	N/A		0.958		Errors: 0.00E+00 S balance		
Heat Exchanger 1	N/A		0.386		-0.00000	0.00000	0.00000
Heat Exchanger 2	N/A		0.488		0.00000	-0.00000	-0.00000
Heat Exchanger 3	N/A		0.664		-0.00000	0.00000	0.00000
Heat Exchanger 4	N/A		0.000		0.00000	above error	

Table 4.24

Temperature at S5  
(T<sub>S5</sub>=450 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	S02
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1146.72	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1094.85	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	450.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	450.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	450.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	450.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	470.34	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	470.34	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	470.34	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	470.34	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.58	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	0.00E+00	S balance	
Heat Exchanger 1	N/A	0.353		-0.00000	0.00000	0.00000	
Heat Exchanger 2	N/A	0.458		0.00000	-0.00000	-0.00000	
Heat Exchanger 3	N/A	0.629		-0.00000	0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.25  
**Temperature at S5**  
 (T<sub>S5</sub>=500 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1189.23	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1137.36	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	500.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	500.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	500.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	500.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	521.46	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	521.46	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	521.46	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	521.46	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.47	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	0.00E+00	S balance	
Heat Exchanger 1	N/A	0.316		-0.00000	0.00000	0.00000	
Heat Exchanger 2	N/A	0.432		0.00000	-0.00000	-0.00000	
Heat Exchanger 3	N/A	0.597		-0.00000	0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.26  
**Temperature at 85**  
**(T<sub>SS</sub>=550 °F)**

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	S02
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1231.74	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1179.87	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	550.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	550.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	550.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	550.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	572.58	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	572.58	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	572.58	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	572.58	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.36	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				

Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	0.00E+00	S balance	
Heat Exchanger 1	N/A	0.275		-0.00000	0.00000	0.00000	
Heat Exchanger 2	N/A	0.408		0.00000	-0.00000	-0.00000	
Heat Exchanger 3	N/A	0.568		-0.00000	0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000C	above error		

Table 4.27

Temperature at S5  
(T<sub>S5</sub>=600 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1274.26	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1222.38	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	600.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	600.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	600.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	600.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	623.70	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	623.70	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	623.70	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	623.70	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.25	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850	N/A			0.500		
Last Stages	0.850	N/A			Mass Flow Rate of Sulfur		
Compressor 2	0.850	N/A			7258 lb/hr		
Intercooler	N/A	0.958			Errors:		
Heat Exchanger 1	N/A	0.229			0.00E+00	S balance	
Heat Exchanger 2	N/A	0.387			-0.00000	0.00000	0.00000
Heat Exchanger 3	N/A	0.540			0.00000	-0.00000	-0.00000
Heat Exchanger 4	N/A	0.000			-0.00000	0.00000	0.00000
					0.00000	above error	



Table 4.28

Temperature at S5  
(T<sub>S5</sub>=650 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	S02
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1316.77	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1264.90	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	650.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	650.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	650.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	650.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	674.82	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	674.82	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	674.82	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	674.82	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.14	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				
Compressor 1:				Percent of Complete Reaction			
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors: 0.00E+00 S balance			
Heat Exchanger 1	N/A	0.177		-0.00000 0.00000 0.00000			
Heat Exchanger 2	N/A	0.368		0.00000 -0.00000 -0.00000			
Heat Exchanger 3	N/A	0.515		-0.00000 0.00000 0.00000			
Heat Exchanger 4	N/A	0.000		0.00000 above error			

Table 4.29  
**Temperature at S5**  
**(T<sub>S5</sub>=700 °F)**

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1359.28	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1307.41	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	700.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	700.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	700.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	700.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	725.93	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	725.93	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	725.93	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	725.93	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	79.03	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.958		Errors:	0.00E+00 S balance		
Heat Exchanger 1	N/A	0.117		-0.00000	0.00000	0.00000	
Heat Exchanger 2	N/A	0.351		0.00000	-0.00000	-0.00000	
Heat Exchanger 3	N/A	0.491		-0.00000	0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Table 4.30

Temperature at S5  
(T<sub>S5</sub>=750 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	SO2
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1401.79	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1349.92	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	750.00	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	750.00	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	750.00	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	750.00	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	777.05	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	777.05	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	777.05	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	777.05	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	78.91	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850	N/A			0.500		
Last Stages	0.850	N/A			Mass Flow Rate of Sulfur		
Compressor 2	0.850	N/A			7258 lb/hr		
Intercooler	N/A	0.958			Errors:		
Heat Exchanger 1	N/A	0.048			0.00E+00 S balance		
Heat Exchanger 2	N/A	0.335			-0.00000 0.00000 0.00000		
Heat Exchanger 3	N/A	0.469			0.00000 -0.00000 -0.00000		
Heat Exchanger 4	N/A	0.000			-0.00000 0.00000 0.00000		
					0.00000 above error		

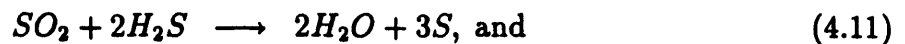
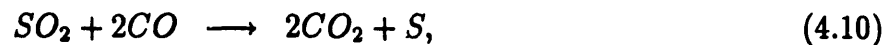
Table 4.31

Temperature at 85  
(T<sub>SS</sub>=780.27 °F)

	Mdot kg/s	Temp deg F	Press psi	Cp KJ/Kg*K	O2	N2	S02
A1	7.41	70.00	14.70	1.0164	0.2100	0.7900	0.0000
A2	7.41	426.18	67.93	1.0164	0.2100	0.7900	0.0000
A3	7.41	85.00	67.93	1.0164	0.2100	0.7900	0.0000
A4	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A5	0.00	451.36	310.00	1.0164	0.2100	0.7900	0.0000
A6	7.41	451.36	314.00	1.0164	0.2100	0.7900	0.0000
A7	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A8	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A9	0.00	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A10	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
A11	3.56	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A12	24.40	802.45	310.00	1.0105	0.0563	0.8430	0.1007
A13	7.41	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A14	3.25	770.00	310.00	1.0164	0.2100	0.7900	0.0000
A15	3.85	770.00	310.00	1.0164	0.2100	0.7900	0.0000
S1	45.82	1427.53	306.00	1.0094	0.0268	0.8532	0.1200
S2	45.82	1427.53	302.00	1.0094	0.0268	0.8532	0.1200
S3	45.82	1375.66	298.00	1.0094	0.0268	0.8532	0.1200
S4	45.82	780.27	294.00	1.0094	0.0268	0.8532	0.1200
S5	7.71	780.27	294.00	1.0094	0.0268	0.8532	0.1200
S6	38.11	780.27	294.00	1.0094	0.0268	0.8532	0.1200
S7	38.11	780.27	294.00	1.0094	0.0268	0.8532	0.1200
S8	38.11	808.00	314.00	1.0094	0.0268	0.8532	0.1200
S9	38.11	808.00	314.00	1.0094	0.0268	0.8532	0.1200
S10	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S11	0.00	808.00	314.00	1.0094	0.0268	0.8532	0.1200
S12	0.00	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S13	38.11	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S14	20.84	808.00	310.00	1.0094	0.0268	0.8532	0.1200
S15	17.27	808.00	310.00	1.0094	0.0268	0.8532	0.1200
W1	60.00	70.00	NC	No Gas			
W2	60.00	80.25	NC	"			
W3	745.00	78.85	NC	"			
W4	745.00	70.00	NC	"			
W5	1.80	70.00	NC	"			
W6	1.80	70.00	NC	"			
Z1	4.89	1000.00	310.00	"			
Z1A	22.16	1249.36	314.00	0.9500	0.0000	0.8126	0.1296
Z1B	46.56	1249.36	314.00	0.9850	0.0326	0.8302	0.1129
Z1C	46.56	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z1D	0.74	1449.31	314.00	0.9939	0.0076	0.8472	0.1306
Z2	4.58	900.00	314.00				
Compressor 1:					Percent of Complete Reaction		
First Stages	0.850	N/A		0.500			
Last Stages	0.850	N/A		Mass Flow Rate of Sulfur			
Compressor 2	0.850	N/A		7258 lb/hr			
Intercooler	N/A	0.953		Errors:	0.00E+00	S balance	
Heat Exchanger 1	N/A	0.000		-0.00000	0.00000	0.00000	
Heat Exchanger 2	N/A	0.326		0.00000	-0.00000	-0.00000	
Heat Exchanger 3	N/A	0.456		-0.00000	0.00000	0.00000	
Heat Exchanger 4	N/A	0.000		0.00000	above error		

Experimental tests at RTI indicate that over 90 percent of the direct sulfur removal can be achieved with this procedure. Higher percentages may be obtained by placing this system in series. For example, if 99% removal is desired, two systems may be placed together.

Under Research Triangle Institute's design, stoichiometric conditions are favorable for the best sulfur producing reaction. As an example [22].



Therefore, a reducing stream of coal gas will be needed for these reactions. With a sulfur percentage of about 4.04 % weight in the inlet coal, about 4.68% of the gasification gas (3.80 kg/sec) will need to be sent to the first direct sulfur recovery unit in order to achieve stoichiometric conditions. If two consecutive units are placed in series to achieve the 99 percent removal goal, a secondary stream of coal gas may need to be bled off. However, this stream should be at least an order of magnitude less than the first.

Table 4.33 is an example of flow rates and mole fractions that may occur in one stage of a direct sulfur recovery plant. Again, it can be seen that a mass flow rate of 3.8 kg/sec of coal gas is needed for a 12%  $SO_2$  stream at 7.71 kg/sec. This preliminary direct sulfur recovery balance is done assuming that the reactions are thermally neutral.

Table 4.33

Direct sulfur Recovery

	S5	Coal Gas Stream	S16	S17	S18	W7	W8
Mass Flow Rate (kg/sec)	7.71	3.80	11.51	11.51	10.66	NC	NC
Temp (F)	780.27	1020.00	859.40	320.00	320.00	70.00	NC
Pressure	NC	NC	NC	NC	NC	NC	NC
y S 0.000043 Psat @ 320 F / P							
Mole Fraction							
O2	0.0268	0.0000	0.0158	0.0158	0.0169		
N2	0.8532	0.3851	0.6599	0.6599	0.7064		
SO2	0.1200	0.0000	0.0071	0.0071	0.0076		
CO	0.0000	0.1393	0.0057	0.0057	0.0061		
H2	0.0000	0.2003	0.0082	0.0082	0.0088		
CO2	0.0000	0.1134	0.0976	0.0976	0.1044		
H2O	0.0000	0.1215	0.1256	0.1256	0.1345		
CH4	0.0000	0.0333	0.0136	0.0136	0.0146		
C2H6	0.0000	0.0008	0.0003	0.0003	0.0003		
H2S	0.0000	0.0063	0.0003	0.0003	0.0003		
Ele S	0.0000	0.0000	0.0659	0.0659	0.0001		
Mass in one Mole (kg)							
O2	0.8576	0.0000	0.5054	0.5054	0.5410		
N2	23.9067	10.7905	18.4912	18.4912	19.7942		
SO2	7.6872	0.0000	0.4528	0.4528	0.4847		
CO	0.0000	3.9019	0.1594	0.1594	0.1707		
H2	0.0000	0.4038	0.0165	0.0165	0.0177		
CO2	0.0000	4.9908	4.2942	4.2942	4.5968		
H2O	0.0000	2.1889	2.2634	2.2634	2.4229		
CH4	0.0000	0.5342	0.2183	0.2183	0.2337		
C2H6	0.0000	0.0241	0.0098	0.0098	0.0105		
H2S	0.0000	0.2147	0.0088	0.0088	0.0094		
Total:	32.4515	23.0490	26.4200	26.4200	28.2816		
Mole Flow Rate (mole/sec)							
O2	0.0064	0.0000	0.0064	0.0064	0.0064		
N2	0.2028	0.0635	0.2663	0.2663	0.2663		
SO2	0.0285	0.0000	0.0029	0.0029	0.0029		
CO	0.0000	0.0230	0.0023	0.0023	0.0023		
H2	0.0000	0.0330	0.0033	0.0033	0.0033		
CO2	0.0000	0.0187	0.0394	0.0394	0.0394		
H2O	0.0000	0.0200	0.0507	0.0507	0.0507		
CH4	0.0000	0.0055	0.0055	0.0055	0.0055		
C2H6	0.0000	0.0001	0.0001	0.0001	0.0001		
H2S	0.0000	0.0010	0.0001	0.0001	0.0001		
Ele S	0.0000	0.0000	0.0266	0.0266	0.000043		
Total:	0.2377	0.1649	0.4036	0.4036	0.3770		

Elemental Balance Check (Error)

S	-8.3E-19
O	-8.7E-19
C	-8.7E-19
H	-1.7E-18
N	-5.6E-17

Sulfur Removed  
0.0266 mole/sec  
0.8517 kg/sec

Sulfur NOT Removed  
0.095980 kg/sec

89.87161 Percent Eff.

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## APPENDICES – FORTRAN PROGRAM SOURCE CODES

- A) IGCC System Model
- B) Graphics Program
- C) Fixed Bed Coal Gasifier Model
- D) Coal Gasifier Devolatilization Subroutines
- E) Tridiagonal Matrix Solver Subroutine
- F) Packed Bed Convective Heat Transfer Coefficient Subroutine
- G) HRSG Bottoming Cycle Subroutines





```

C
C
DO 10 J=1,12
  ID(J)=J
10 CONTINUE
C
OPEN(UNIT=13, FILE='DATA', STATUS='UNKNOWN')
ITFACT=1
ITEST=0
ICPAIR=1.0035
IKAIR=1.400
ITAMB=298.15
IPAMB=101.325
IMDOTAIR=633.*.4536
IVINLET=10.
ITCEFFT=.92
ITCEFFC=.86
IPEFF=.95
ICOMEFF=.95
IGASEFF=.96
IGENEFF=.96
IMSFRACT=.088
IFCOOL=.05
IPRESSGAS=2068.428
IMC=81.3d0
IMH2=5.3d0
IMO2=9.8d0
IMN2=1.7d0
IMS=1.9d0
INHVCOAL=33725.0d0
IPERH2O=.10d0
IPERASH=.10d0
IFLAG2=0
IZ1=.5
IZ2=.215
INN2=.435d0
INCO2=.103d0
INCO=.212d0
INH2=.201d0
INCN4=.046d0
INC2H6=.003d0
C
98 CONTINUE
DO 799 K=1,2
  IF(K.EQ.1) L=6
  IF(K.EQ.2) L=13
DO 97 I=1,8
  WRITE(L,*) ' '
97 CONTINUE
WRITE(L,*) ' *****
WRITE(L,*) ' ** COMMAND MENU **
WRITE(L,*) ' *****
WRITE(L,*) '
WRITE(L,*) ' Please type the number or letter of the variable'
WRITE(L,*) ' you wish to change and PRESS ENTER.'
WRITE(L,*) '
WRITE(L,*) ' -----
WRITE(L,*) ' (TOGGLE VALUES)'
WRITE(L,301)
IF(ITEST.EQ.0) THEN
  WRITE(L,*) ' COALGAS'
ENDIF
IF(ITEST.EQ.1) THEN
  WRITE(L,*) ' METHANE'
  IMSFRACT=0.
  INCO2=0.

```

```

      INCO=0.
      INH2=0.
      INCH4=1.
      INC2H6=0.
      INN2O=0.
      INN2=0.
      INH2S=0.
      INSO2=0.
      INO2=0.
ENDIF
IF(ITEST.EQ.0) THEN
  WRITE(L,302)
  IF(IFLAG2.EQ.0) THEN
    WRITE(L,*) '
                                EXPERIMENTAL'
  ENDIF
  IF(IFLAG2.EQ.1) THEN
    WRITE(L,*) '
                                THEORETICAL -> (EQUILIBRIUM)'
  ENDIF
ENDIF
WRITE(L,*) '
-----'
WRITE(L,*) '
                                (INPUT VALUES)'
WRITE(L,303) ITAMB
WRITE(L,304) IPAMB
WRITE(L,305) IMDOTAIR
WRITE(L,306) IFCOOL
WRITE(L,307) ITCEFFC
WRITE(L,308) ITCEFFT
WRITE(L,309) ICOMEFF
WRITE(L,310) IPEFF
WRITE(L,311) IGENEFF
IF(ITEST.EQ.0) THEN
  WRITE(L,401) IGASEFF
  WRITE(L,402) IPRESSGAS
  WRITE(L,403) INSFRACT
  WRITE(L,*) ' F)
  WRITE(L,*) '
                                COAL STATISTICS'
                                (Amounts in MASS FRACTIONS)'
  WRITE(L,404) INC
  WRITE(L,405) INH2
  WRITE(L,406) INO2
  WRITE(L,407) INN2
  WRITE(L,408) INS
  WRITE(L,409) INHVCOAL
  WRITE(L,410) IPERN2O
  WRITE(L,411) IPERASH
  WRITE(L,412) IZ1
ENDIF
IF(IFLAG.EQ.0) THEN
  WRITE(L,*) ' !)
  WRITE(L,*) '
                                PRODUCT STATISTICS'
                                (Amounts in moles - DRY & Sulfur FREE)'
  WRITE(L,414) INCO2
  WRITE(L,415) INCO
  WRITE(L,416) INH2
  WRITE(L,417) INCH4
  WRITE(L,418) INC2H6
  WRITE(L,419) INN2
ENDIF
IF(IFLAG.EQ.1) THEN
  WRITE(L,413) IZ2
ENDIF
IF(ITEST.EQ.0) THEN
  WRITE(L,*) ' '
  WRITE(L,505)
  IF(ITFACT.EQ.1) THEN
    WRITE(L,507)
  ENDIF
ENDIF

```

```

      IF(ITFACT.EQ.0) THEN
        WRITE(L,506)
      ENDIF
    ENDIF
  C
  C
301  FORMAT(54H 1)           Fuel used in power system           )
302  FORMAT(54H 2)           Method of finding products leaving gasifier )
303  FORMAT(54H 3) Temperature of air at inlet (Location 1)..... ,
    & F6.2,2H K)
304  FORMAT(54H 4) Pressure of air at inlet (Location 1)..... ,
    & F6.2,4H KPA)
305  FORMAT(53H 5) Mass flow rate of air at inlet (Location 1)... ,
    & F7.2,7H KG/SEC)
306  FORMAT(54H 6) Fraction of air used to cool blades..... ,
    & F5.3)
307  FORMAT(54H 7) Efficiency of air compressors..... ,
    & F5.3)
308  FORMAT(54H 8) Efficiency of gas turbines..... ,
    & F5.3)
309  FORMAT(54H 9) Efficiency of combustor..... ,
    & F5.3)
310  FORMAT(54H A) Efficiency of pipes..... ,
    & F5.3)
311  FORMAT(54H B) Efficiency of generator and drive..... ,
    & F5.3)
401  FORMAT(54H C) Efficiency of gasifier..... ,
    & F5.3)
402  FORMAT(52H D) Pressure in gasifier..... ,
    & F9.2,4H KPA)
403  FORMAT(54H E) Fraction of air sent to gasifier..... ,
    & F5.3)
404  FORMAT(33H              Carbon.... ,F7.3)
405  FORMAT(33H              Hydrogen... ,F7.3)
406  FORMAT(33H              Oxygen.... ,F7.3)
407  FORMAT(33H              Nitrogen... ,F7.3)
408  FORMAT(33H              Sulfur.... ,F7.3)
409  FORMAT(41H              Heating value of coal... ,F7.1)
410  FORMAT(49H              Percent WATER in coal as received... ,
    & F5.3)
411  FORMAT(48H              Percent ASH in coal as received... ,F5.3)
412  FORMAT(54H G) Amount of water added per mole of dry coal.... ,
    & F5.3)
413  FORMAT(54H H) Amount of air added per mole of dry coal..... ,
    & F5.3)
414  FORMAT(31H              CO2.... ,F8.5)
415  FORMAT(31H              CO..... ,F8.5)
416  FORMAT(31H              N2..... ,F8.5)
417  FORMAT(31H              CH4.... ,F8.5)
418  FORMAT(31H              CnHm... ,F8.5)
419  FORMAT(31H              N2..... ,F8.5)
503  FORMAT(40H              Please input the new value ..... )
505  FORMAT(41H J)              AUXILIARY TURBINE STATUS)
506  FORMAT(41H              ---> THROTTLE <--- )
507  FORMAT(44H              ---> AUXILIARY TURBINE <--- )
    WRITE(L,*) ' '
    WRITE(L,*) '
    WRITE(L,*) '              ----- INPUT SELECTION <-----'
    WRITE(L,*) '              (ENTER -> 0 <- TO CONTINUE ONWARD)'
    WRITE(L,*) ' '
799  CONTINUE
    READ (5,'(A1)') QUEST
    IF(QUEST.EQ.'0') GO TO 96
    IF(QUEST.EQ.'1') THEN
      EXIT=0
      IF(ITEST.EQ.0) THEN

```

```

      ITEST=1
      EXIT=1
    ENDIF
    IF(ITEST.EQ.1.AND.EXIT.EQ.0) THEN
      ITEST=0
    ENDIF
  ENDIF
  IF(QUEST.EQ.'2') THEN
    EXIT=0
    IF(IFLAG2.EQ.0) THEN
      IFLAG2=1
      EXIT=1
    ENDIF
    IF(IFLAG2.EQ.1.AND.EXIT.EQ.0) THEN
      IFLAG2=0
    ENDIF
  ENDIF
  IF(QUEST.EQ.'3') THEN
    WRITE(6,503)
    READ(5,*) ITAMB
  ENDIF
  IF(QUEST.EQ.'4') THEN
    WRITE(6,503)
    READ(5,*) IPAMB
  ENDIF
  IF(QUEST.EQ.'5') THEN
    WRITE(6,503)
    READ(5,*) IMDOTAIR
  ENDIF
  IF(QUEST.EQ.'6') THEN
    WRITE(6,503)
    READ(5,*) IFCOOL
  ENDIF
  IF(QUEST.EQ.'7') THEN
    WRITE(6,503)
    READ(5,*) ITCEFFC
  ENDIF
  IF(QUEST.EQ.'8') THEN
    WRITE(6,503)
    READ(5,*) ITCEFFT
  ENDIF
  IF(QUEST.EQ.'9') THEN
    WRITE(6,503)
    READ(5,*) ICOMEFF
  ENDIF
  IF(QUEST.EQ.'A') THEN
    WRITE(6,503)
    READ(5,*) IPEFF
  ENDIF
  IF(QUEST.EQ.'B') THEN
    WRITE(6,503)
    READ(5,*) IGENEFF
  ENDIF
  IF(QUEST.EQ.'C') THEN
    WRITE(6,503)
    READ(5,*) IGASEFF
  ENDIF
  IF(QUEST.EQ.'D') THEN
    WRITE(6,503)
    READ(5,*) IPRESSGAS
  ENDIF
  IF(QUEST.EQ.'E') THEN
    WRITE(6,503)
    READ(5,*) IMSFRACT
  ENDIF

```

```

IF(QUEST.EQ.'F') THEN
  WRITE(6,*)'
  READ(5,*) IMC
  WRITE(6,*)'
  READ(5,*) IMH2
  WRITE(6,*)'
  READ(5,*) IMO2
  WRITE(6,*)'
  READ(5,*) IMN2
  WRITE(6,*)'
  READ(5,*) IMS
ENDIF
IF(QUEST.EQ.'G') THEN
  WRITE(6,503)
  READ(5,*) I21
ENDIF
IF(QUEST.EQ.'H') THEN
  WRITE(6,503)
  READ(5,*) I22
ENDIF
IF(QUEST.EQ.'I') THEN
  WRITE(6,*)'
  READ(5,*) INCO2
  WRITE(6,*)'
  READ(5,*) INCO
  WRITE(6,*)'
  READ(5,*) INH2
  WRITE(6,*)'
  READ(5,*) INCH4
  WRITE(6,*)'
  READ(5,*) INC2H6
  WRITE(6,*)'
  READ(5,*) INN2
ENDIF
IF(QUEST.EQ.'J') THEN
  EXIT=0
  IF(ITFACT.EQ.0) THEN
    ITFACT=1
    EXIT=1
  ENDIF
  IF(ITFACT.EQ.1.AND.EXIT.EQ.0) THEN
    ITFACT=0
  ENDIF
ENDIF
GO TO 98
96 CONTINUE
WRITE(6,*) 'Is a desulfurization plant in'
READ (5, '(A1)') QS
C
COUNT2=1
MDOTA(1)=633.*.4536*.99
MDOTA(2)=633.*.4536
902 COUNT2=COUNT2+1
DO 901 I2=1,2
  TFACT=ITFACT
  TEST=ITEST
  CPAIR=ICPAIR
  KAIR=KPAIR
  TAMB=ITAMB
  PAMB=IPAMB
  MDOAIR=MDOTA(I2)
  VINLET=IVINLET
  TCEFFT=ITCEFFT
  TCEFFC=ITCEFFC
  PEFF=IPEFF

```

Input the new amount of CARBON.....'

Input the new amount of HYDROGEN...'

Input the new amount of OXYGEN.....'

Input the new amount of NITROGEN...'

Input the new amount of SULFUR.....'

Input the new amount of CO2.....'

Input the new amount of CO.....'

Input the new amount of H2.....'

Input the new amount of CH4.....'

Input the new amount of CnHm...'

Input the new amount of N2.....'



```

CONEFF=ICONEFF
GASEFF=IGASEFF
GENEFF=IGENEFF
MSFRACT=MSFRACT
FCOOL=IFCOOL
PRESSGAS=IPRESSGAS
MC=IMC
MH2=IMH2
MO2=IMO2
MN2=IMN2
MS=IMS
HHVCOAL=IHHVCOAL
PERH2O=IPERH2O
PERASH=IPERASH
FLAG2=IFLAG2
Z1=IZ1
Z2=IZ2
NN2=INN2
NCO2=INCO2
NCO=INCO
NH2=INH2
NCH4=INCH4
NC2H6=INC2H6
NH2O=0.
NH2S=0.
NSO2=0.
NO2=0.

```

```

C *****
C ** PARAMETER ASSIGNMENT **
C *****

```

```

PIPEFF(1)=PEFF
PIPEFF(2)=PEFF
PIPEFF(3)=PEFF
PIPEFF(4)=PEFF
PIPEFF(5)=PEFF
PIPEFF(6)=PEFF
NPC1=TCEFFC
NPC2=TCEFFC
NPT1=TCEFFT*TFAC
NPT2=TCEFFT
P3P2=11.7

```

```

C *****
C ** MAIN SYSTEM **
C *****

```

```

C --> INPUT INITIAL CONDITIONS <--
T(1)=TAMB
P(1)=PAMB
MDOT(1)=MDOTAIR

```

```

C --> 1 TO 2 (ADJUSTING FOR INCREASE IN SPEED OF GAS) <--
V(2)=VINLET
T(2)=T(1)-(V(2)**2./(2000.*CPAIR))
P(2)=P(1)*(T(2)/T(1))**(KAIR/(KAIR-1.))
MDOT(2)=MDOT(1)

```

```

C --> 2 TO 3 (ACROSS FIRST COMPRESSOR) <--
RPM=3600.
X=MDOTA(12)/122.55
Y=RPM/(RPM/.9979)
XO=2.33*Y**3.
P3P2=1.+7.5*(X/Y)**2.-(X/(Y*Y))**4.
NPC1=.5125005*(1.+25*DEXP(-25.*(Y-.9)**2.))+.5*
& DEXP(-50.*(X-XO)**2.)
V(3)=V(2)
T(3)=T(2)*P3P2**((KAIR-1.)/(NPC1*KAIR))
DO 589 I=1,3
TAVG=(T(2)+T(3))/2.

```

```

CP1=32.901-1434.17/TAVG
CP2=36.599-2521.26/TAVG
CPA=(CP2+3.77*CP1)/(4.77*28.97)
CV=CPA-.287
KA=CPA/CV
T(3)=T(2)*P3P2**((KA-1.)/(NPC1*KA))
WCOMP1=MDOT(2)*CPA*(T(2)-T(3))
589 CONTINUE
P(3)=P3P2*P(2)
MDOT(3)=MDOT(2)
C --> 3 TO 4 (PRESSURE LOSS IN PIPES) <--
V(4)=V(3)
T(4)=T(3)
P(4)=PIPEFF(1)*P(3)
MDOT(4)=MDOT(3)
C --> 4 TO 13 (DIVISION OF STEAM TO COMBUSTOR) <--
V(13)=V(4)
T(13)=T(4)
P(13)=P(4)
MDOT(13)=(1.-MSFRACT-FCOOL)*MDOT(4)
C --> 13 TO 14 (PRESSURE LOSS IN PIPES) <--
V(14)=V(13)
T(14)=T(13)
P(14)=PIPEFF(6)*P(13)
MDOT(14)=MDOT(13)
C *** branch ***
IF(TEST.EQ.1) GO TO 14
C --> 4 TO 5 (DIVISION OF STEAM TO SECOND COMPRESSOR) <--
V(5)=V(4)
T(5)=T(4)
P(5)=P(4)
MDOT(5)=MSFRACT*MDOT(4)
C --> 5 TO 6 (PRESSURE LOSS IN PIPES) <--
V(6)=V(5)
T(6)=T(5)
P(6)=PIPEFF(2)*P(5)
MDOT(6)=MDOT(5)
C --> 6 TO 7 TO 8 (ACROSS SECOND COMPRESSOR & LOSS IN PIPES) <--
P(8)=PRESSGAS
P(7)=P(8)/PIPEFF(3)
V(7)=V(6)
T(7)=T(6)*(P(7)/P(6))**((KAIR-1.)/(NPC2*KAIR))
DO 588 I=1,3
TAVG=(T(6)+T(7))/2.
CP1=32.901-1434.17/TAVG
CP2=36.599-2521.26/TAVG
CPA=(CP2+3.77*CP1)/(4.77*28.97)
CV=CPA-.287
KA=CPA/CV
T(7)=T(6)*(P(7)/P(6))**((KA-1.)/(NPC2*KA))
WCOMP2=MDOT(6)*CPA*(T(6)-T(7))
588 CONTINUE
MDOT(7)=MDOT(6)
V(8)=V(7)
T(8)=T(7)
MDOT(8)=MDOT(7)
C --> 8 TO 9 (GASIFIER) <--
V(9)=V(8)
TAIR=T(8)
TLIQ=25.d0
TSTEAM=573.15d0
MDOT(21)=.083744*21*MDOT(8)/Z2
MDOT(22)=.5625*MDOT(21)
IF(FLAG2.EQ.0) THEN
CALL GASTEMP(MC,MH2,MO2,MN2,MS,PERH2O,PERASH,HHVCOAL,NN2,NO2,

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& NCO2,NCO,NH2O,NH2,NCH4,NC2H6,NSO2,NH2S,NOH,NNO,MLIQH2O,
& TLIQ,NSTEAM,TSTEAM,GASEFF,TPROD,Z1,Z2,TAIR,MASH,MH2O)
ENDIF
IF(FLAG2.EQ.1) THEN
CALL GASIFIER(NC,MH2,MO2,MN2,MS,PERH2O,PERASH,HHVCOAL,NN2,NO2,
& NCO2,NCO,NH2O,NH2,NCH4,NC2H6,NSO2,NH2S,NOH,NNO,MLIQH2O,
& TLIQ,NSTEAM,TSTEAM,GASEFF,TPROD,Z1,Z2,TAIR,MASH,MH2O)
ENDIF
NCO2G=NCO2
NCOG=NCO
NH2G=NH2
NCH4G=NCH4
NC2H6G=NC2H6
NN2G=NN2
MH2OG=MH2O
NH2SG=NH2S
T(20)=25.0
T(21)=TSTEAM
T(22)=TLIQ
T(9)=TPROD
NC1=NC/12.01
NH21=NH2/2.016
NO21=NO2/32.000
NN21=NN2/28.013
NS1=NS/32.06
SUMN=NC1+NH21+NO21+NN21+NS1
NC1=NC1/SUMN
NH21=NH21/SUMN
NO21=NO21/SUMN
NN21=NN21/SUMN
NS1=NS1/SUMN
MCOAL=NC1*12.011+NH21*2.016+NO21*31.999+NN21*28.013+NS1*32.064
LHVCOAL=HHVCOAL-(NH21-NS1)*44000./MCOAL
MGAS=(NCO*28.011+NCO2*44.01+NH2*2.016+MH2O*18.015+NN2*28.013+
& NH2S*34.0760+NCH4*16.043+NC2H6*30.070+NSO2*64.0588)/
& (NCO+NCO2+NH2+MH2O+NN2+NH2S+NCH4+NC2H6+NSO2)
MASSGAS=NCO*28.011+NCO2*44.01+NH2*2.016+MH2O*18.015+NN2*28.013+
& NH2S*34.0760+NCH4*16.043+NC2H6*30.070+NSO2*64.0588
MASSCOAL=MCOAL+MASH+MH2O
MASSAIR=Z2*(3.77*28.013+32.0)
MASSH2O=Z1*18.015
RGAS=1545./MGAS
LVGAS=(RGAS*536.67)/(14.6959*144.)
LVAIR=(53.3*536.67)/(14.6959*144.)
V1=(LVGAS*MASSGAS)/MASSCOAL
V2=MASSH2O/(LVGAS*MASSGAS)
V3=(LVAIR*MASSAIR)/(LVGAS*MASSGAS)
LHVCOAL=HHVCOAL-NH21*44011/MCOAL
MDOT(20)=MCOAL*MDOT(8)/(137.56*Z2)
DO 11 I=1,12
CP(I)=A(I)-B(I)/900.
11 CONTINUE
CALL GASPROP(NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NN2,NH2S,NSO2,NO2
& ,CPPROD,MPROD,RPROD,CVPROD,KPROD,CP)
HHVGAS = (NCH4*((-74873)-(-393522))-
& 2.*(-285838))+NC2H6*((-84667)-2.*(-393522))-3.*(-285838))+
& NH2*(0.-(-285838))+NCO*((-110529)-(-393522))+
& NH2S*((-20142)-(-285838)-(-296847)))/(NCH4+NC2H6+
& NH2+NCO+NH2S+NCO2+NN2)
HHVGAS=HHVGAS/MPROD
LHV GAS = (NCH4*((-74873)-(-393522))-
& 2.*(-241827))+NC2H6*((-84667)-2.*(-393522))-3.*(-241827))+
& NH2*(0.-(-241827))+NCO*((-110529)-(-393522))+
& NH2S*((-20142)-(-241827)-(-296847)))/(NCH4+NC2H6+NH2+NCO+
& NH2S+NCO2+NN2)

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LHV GAS=LHV GAS/MPROD
HHV2D=NCO*282993.+NH2*285838.+NCH4*890322.+NC2H6*1559881.+
& NH2S*562543.+NH2O*44000.
LHV2D=NCO*282993.+NH2*241827.+NCH4*802310.+NC2H6*1427844.+
& NH2S*518543.
M1D=(NCO*28.01+NCO2*44.01+NH2*2.016+NCH4*16.043+NC2H6*30.07+
& NH2*28.013+NH2S*34.22)
M2D=M1D/(NCO+NCO2+NH2+NCH4+NC2H6+NH2+NH2S)
HHVPD=HHV2D/M1D
LHVPD=LHV2D/M1D
HHVPPD=(1.1127D-3)*M2D*HHVPD
LHVPPD=(1.1127D-3)*M2D*LHVPD
HHV2W=NCO*282993.+NH2*285838.+NCH4*890322.+NC2H6*1559881.+
& NH2S*562543.+NH2O*44000.
LHV2W=NCO*282993.+NH2*241827.+NCH4*802310.+NC2H6*1427844.+
& NH2S*518543.
M1W=(NCO*28.01+NCO2*44.01+NH2*2.016+NCH4*16.043+NC2H6*30.07+
& NH2*28.013+NH2O*18.015+NH2S*34.22)
M2W=M1W/(NCO+NCO2+NH2+NCH4+NC2H6+NH2+NH2O+NH2S)
HHVPW=HHV2W/M1W
LHV PW=LHV2W/M1W
HHVPPW=(1.1127D-3)*M2W*HHVPW
LHVPPW=(1.1127D-3)*M2W*LHV PW
P(9)=PRESSGAS
MDOTASH=((PERASH*MDOT(20))/(1.-PERASH)+(PERASH*PERH2O*MDOT(20))/
& ((1.-PERH2O)*(1.-PERASH)))/(1.-((PERASH*PERH2O)/((1.-PERASH)*
& (1.-PERH2O))))
MDOTH2O=(PERH2O*MDOT(20)+PERH2O*MDOTASH)/(1.-PERH2O)
MDOT(23)=MDOTASH
MDOT(9)=MDOT(8)+MDOT(20)+MDOT(21)+MDOT(22)+MDOTH2O
MDOT(20)=MDOT(20)+MDOTASH+MDOTH2O
C --> 9 TO 10 (PRESSURE LOSS IN PIPES) <--
V(10)=V(9)
T(10)=T(9)
P(10)=PIPEFF(4)*P(9)
MDOT(10)=MDOT(9)
C --> 10 TO 11 TO 12 (ACROSS FIRST TURBINE & LOSS IN PIPES) <--
P(12)=P(14)
P(11)=P(12)/PIPEFF(5)
V(11)=V(10)
T(11)=T(10)*(P(11)/P(10))**((NPT1*(KPROD-1.))/KPROD)
MDOT(11)=MDOT(10)
WTURB1=MDOT(11)*CPPROD*(T(10)-T(11))
V(12)=V(11)
T(12)=T(11)
MDOT(12)=MDOT(11)
C --> 12 & 14 TO 15 (THROUGH COMBUSTOR) <--
Z=MDOT(13)*(NCO*28.011+NCO2*44.01+NH2*2.016+NH2O*18.015+
& NH2*28.013+NH2S*34.0760+NCH4*16.043+NC2H6*30.070+
& NSO2*64.0588)/(137.61*MDOT(11))
XMSO2=(NCO*28.011+NCO2*44.01+NH2*2.016+NH2O*18.015+
& NH2*28.013+NH2S*34.0760+NCH4*16.043+NC2H6*30.070+
& NSO2*64.0588)
14 IF( TEST.EQ.1) THEN
Z=6.37
P(12)=P(14)
P(11)=150.
T(11)=300.
V(11)=V(2)
MDOT(11)=MDOT(13)*.1166/Z
DO 13 I=1,12
CP(I)=A(I)-B(I)/450.
13 CONTINUE
CALL GASPROP(NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NH2,NH2S,NSO2,NO2
& ,CPPROD,MPROD,RPROD,CVPROD,KPROD,CP)

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V(12)=V(11)
T(12)=T(11)*(P(12)/P(11))**((KPROD-1.)/(1.9*KPROD))
MDOT(12)=MDOT(11)
WCOMP2=MDOT(12)*CPPROD*(T(11)-T(12))
ENDIF
TAIR=T(14)
TREAT=T(12)
PRESS=P(14)
LHV=LHVGAS
IF(TEST.EQ.1) THEN
  LHV=50010.
ENDIF
IF(QS.EQ.'Y'.OR.QS.EQ.'y') NH2S=NH2S*.050
CALL COMBUSTOR(NH2,Z,NCO2,NCO,NH2O,NH2,NCH4,NC2H6,
& NSO2,NH2S,TAIR,TREAT,TPROD,PRESS,N,NO2,LHV,COMEFF)
V(15)=V(14)
T(15)=TPROD
P(15)=P(14)
MDOT(15)=MDOT(14)+MDOT(12)
N=7
C --> 15 TO 16 (ACROSS SECOND TURBINE)
DO 12 I=1,12
  CP(I)=A(I)-B(I)/1200.
12 CONTINUE
CALL GASPROP(NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NH2,NH2S,NSO2,NO2
& ,CPPROD,NPROD,RPROD,CVPROD,KPROD,CP)
P(18)=102.000
P(16)=(P(18)/P(15))**.3333333333*P(15)
KT=(277.77*(T(15)**.5))/P(15)
MDOTT(12)=((KT*P(15))/(T(15)**.5))
DIFF2(12)=MDOTT(12)-MDOT(15)
901 CONTINUE
MDOTA(2)=(MDOTA(2)*DIFF2(1)-MDOTA(1)*DIFF2(2))/
& (DIFF2(1)-DIFF2(2))
MDOTA(1)=.99*MDOTA(2)
TEST2=ABS(DIFF2(2))
IF(TEST2.GT..001) GO TO 902
P(17)=P(16)
V(16)=V(15)
T(16)=T(15)*(P(16)/P(15))**((NPT2*(KPROD-1.))/KPROD)
MDOT(16)=MDOT(15)
WTURB21=MDOT(16)*CPPROD*(T(15)-T(16))
MDOT(17)=MDOT(16)+MDOT(4)*FCOOL
T(17)=(FCOOL*MDOT(4)*T(4)+MDOT(16)*T(16))/(FCOOL*MDOT(4)+
& MDOT(16))
V(17)=V(16)
V(18)=V(17)
T(18)=T(17)*(P(18)/P(17))**((NPT2*(KPROD-1.))/KPROD)
MDOT(18)=MDOT(17)
WTURB22=MDOT(17)*CPPROD*(T(17)-T(18))
WTURB2=WTURB21+WTURB22
C ** SECOND COMBUSTOR **
RAT(1)=.020
RAT(2)=.018
829 COUNTR=COUNTR+1.
DO 849 I=1,2
  HHVC=HHVCOAL
  TREAT=T(18)
  TPROD=0.
  PRESS=P(18)
  NC1=NC*RAT(1)/12.01
  NH21=NH2*RAT(1)/2.016
  NO21=NO2*RAT(1)/32.000
  NH21=NH2*RAT(1)/28.013
  HS1=HS*RAT(1)/32.06

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MCF=MC1
MH2F=MH21
MO2F=MO21
MH2F=MH21
MSF=MS1
NH2S=NH2
NO2S=NO2
NCO2S=NCO2
NCOS=NCO
NH2OS=NH2O
NH2SE=NH2
NCH4S=NCH4
NC2H6S=NC2H6
NSO2S=NSO2
NH2SS=NH2S
NOHS=NOH
NNOS=NNO
MFINES=MCF*12.011+MH2F*2.016+MO2F*31.999+MH2F*28.013+
& MSF*32.064
MGAS=NH2S*28.013+NO2S*32.000+NCO2S*44.010+NCOS*28.010+
& NH2OS*18.015+NH2SE*2.016+NCH4S*16.043+NC2H6S*28.054+
& NSO2S*64.066+NH2SS*34.076+NOHS*17.007+NNOS*30.006
PEREXTRAS=PERH2O+PERASH
MEXTRAS=(PEREXTRAS*MFINES)/(1.-PEREXTRAS)
MASH=(PERASH/PEREXTRAS)*MEXTRAS
MH2O=(PERH2O/PEREXTRAS)*MEXTRAS
NH2OS=MH2OS+MH2O/18.015
MAIR=MGAS*FCOOL*MDOT(4)/MDOT(15)
NO2S=NO2S+MAIR/137.56
NH2S=NH2S+MAIR*3.77/137.56
CALL COMBUSTOR2(NH2S,NO2S,NCO2S,NCOS,NH2OS,NH2SS,NCH4S,NC2H6S,
& NSO2S,NH2SS,NNOS,NOHS,MCF,MH2F,MO2F,MH2F,MSF,MHVC,MASH,
& TREAT,TPROD,PRESS,COMEFF,MFINES,DIFFH)
DIFFY(1)=DIFFH
849 CONTINUE
RAT(2)=(RAT(2)*DIFFY(1)-RAT(1)*DIFFY(2))/(DIFFY(1)-DIFFY(2))
RAT(1)=1.05*RAT(2)
TEST2=ABS(DIFFY(2))
IF(TEST2.GT..1) GO TO 829
MDOT(50)=MDOT(15)*MFINES/MGAS
T(50)=298.15
P(50)=101.325
MDOT(51)=MDOT(15)*MASH/MGAS
T(51)=TPROD
P(51)=PRESS
MDOT(19)=MDOT(50)+MDOT(18)-MDOT(51)
T(19)=TPROD
P(19)=P(18)
MDOTASH2=((PERASH*MDOT(50))/(1.-PERASH)+(PERASH*PERH2O*MDOT(50))/
& ((1.-PERH2O)*(1.-PERASH)))/(1.-((PERASH*PERH2O)/((1.-PERASH)*
& (1.-PERH2O))))
MDOTH2O2=(PERH2O*MDOT(50)+PERH2O*MDOTASH2)/(1.-PERH2O)
C
IF (TEST.EQ.1) THEN
TOT2WORK=WTURB2+WCOMP1
TOTWORK=TOT2WORK
ENDIF
IF (TEST.EQ.0) THEN
TOT1WORK=WTURB1+WTURB2+WCOMP1+WCOMP2
TOT2WORK=WTURB2+WCOMP1+WCOMP2
IF(TFACT.EQ.0) THEN
TOTWORK=TOT2WORK
ENDIF
IF(YFACT.EQ.1) THEN
TOTWORK=TOT1WORK

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ENDIF
T(23)=1000.
P(20)=P(8)
P(21)=P(8)
P(22)=P(8)
P(23)=P(8)
ENDIF
T(20)=T(20)+273.15
T(22)=T(22)+273.15
TOTWORK=TOTWORK*GENEFF
TOT1WORK=TOT1WORK*GENEFF
TOT2WORK=TOT2WORK*GENEFF
C
CALL GASPROP(NCO2, NCO, NH2, NCH4, NC2H6, NN2O, NN2, NH2S, NSO2, NO2
& , CPPROD, MPROD, RPROD, CVPROD, KPROD, CP)
CALL HEATRECOVERY(T, P, MDOT, UMET, LPUMPR1, LPUMPR2, LPUMPR3,
& WTURBR1, WTURBR2, WTURBR3, WTURBR4, QSUPER, QHPBOIL, QLPBOIL,
& QCONDENSER, QPIPELOSS, CPPROD)
WTURBR=UMET
TOT3WORK=TOT1WORK+WTURBR
TOT4WORK=TOT2WORK+WTURBR
C
C ** GAS TURBINE - THROTTLE AFTER GASIFIER
EFFGAS2=TOT2WORK/(MDOT(9)*LHVPU)
HRGAS2=3413./EFFGAS2
EFFCOAL2=TOT2WORK/((MDOT(20)-MDOTASH-MDOTH2O)*LHVCOAL)
EFF2=TOT2WORK/(MDOT(12)*50010)
HRCOAL2=3413./EFFCOAL2
HR2=3413./EFF2
EFGAS2=TOT2WORK/(MDOT(9)*HHVPU)
HRGW2=3413./EFGAS2
EFCOAL2=TOT2WORK/((MDOT(20)-MDOTASH-MDOTH2O)*HHVCOAL)
HRC2=3413./EFCOAL2
EFF=EFF2
HR=HR2
C
C ** GAS TURBINE - RECOVERY TURBINE - THROTTLE AFTER GASIFIER **
EFFGAS4=TOT4WORK/(MDOT(9)*LHVPU)
HRGAS4=3413./EFFGAS4
EFFCOAL4=TOT4WORK/((MDOT(20)-MDOTASH-MDOTH2O+MDOT(50)-MDOTASH2-
& MDOTH2O2)*LHVCOAL)
EFF4=TOT4WORK/(MDOT(12)*50010)
HRCOAL4=3413./EFFCOAL4
HR4=3413./EFF4
EFGAS4=TOT4WORK/(MDOT(9)*HHVPU)
HRGW4=3413./EFGAS4
EFCOAL4=TOT4WORK/((MDOT(20)-MDOTASH-MDOTH2O+MDOT(50)-MDOTASH2-
& MDOTH2O2)*HHVCOAL)
HRC4=3413./EFCOAL4
C
C ** GAS TURBINE - TURBINE AFTER GASIFIER **
EFFGAS1=TOT1WORK/(MDOT(9)*LHVPU)
HRGAS1=3413./EFFGAS1
EFFCOAL1=TOT1WORK/((MDOT(20)-MDOTASH-MDOTH2O)*LHVCOAL)
EFF1=TOT1WORK/(MDOT(12)*50010)
HRCOAL1=3413./EFFCOAL1
HR1=3413./EFF1
EFGAS1=TOT1WORK/(MDOT(9)*HHVPU)
HRGW1=3413./EFGAS1
EFCOAL1=TOT1WORK/((MDOT(20)-MDOTASH-MDOTH2O)*HHVCOAL)
HRC1=3413./EFCOAL1
C
C ** GAS TURBINE - RECOVERY TURBINES - TURBINE AFTER GASIFIER **
EFFGAS3=TOT3WORK/(MDOT(9)*LHVPU)
HRGAS3=3413./EFFGAS3

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EFFCOAL3=TOT3WORK/((MDOT(20)-MDOTASH-MDOTH20+MDOT(50)-MDOTASH2-
& MDOTN2O2)*LHVCOAL)
EFF3=TOT3WORK/(MDOT(12)*50010)
HRCOAL3=3413./EFFCOAL3
HR3=3413./EFF3
EFGASL3=TOT3WORK/(MDOT(9)*HHVPW)
HRCGL3=3413./EFGASL3
EFCOAL3=TOT3WORK/((MDOT(20)-MDOTASH-MDOTH20+MDOT(50)-MDOTASH2-
& MDOTN2O2)*HHVCOAL)
HRC3=3413./EFCOAL3

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C
C *****
C ** PRINTING **
C *****
DO 797 K=1,2
IF(K.EQ.1) L=6
IF(K.EQ.2) L=13
IF(FLAG2.EQ.1) THEN
  WRITE(L,*) ' '
  WRITE(L,*) '          GASIFIER PRODUCTS'
  SSUM=NCO2G+NCOG+NH2G+NCH4G+NC2H6G+NN2G+NH2OG+NH2SG
  WRITE(L,701) NCO2G/SSUM
  WRITE(L,702) NCOG/SSUM
  WRITE(L,703) NH2G/SSUM
  WRITE(L,704) NCH4G/SSUM
  WRITE(L,705) NC2H6G/SSUM
  WRITE(L,706) NN2G/SSUM
  WRITE(L,707) NH2OG/SSUM
  WRITE(L,708) NH2SG/SSUM
701  FORMAT(27H          CO2 -----> ,F9.7)
702  FORMAT(27H          CO -----> ,F9.7)
703  FORMAT(27H          H2 -----> ,F9.7)
704  FORMAT(27H          CH4 -----> ,F9.7)
705  FORMAT(27H          C2H6 -----> ,F9.7)
706  FORMAT(27H          N2 -----> ,F9.7)
707  FORMAT(27H          H2O -----> ,F9.7)
708  FORMAT(27H          H2S -----> ,F9.7)
ENDIF
  WRITE(L,*) ' '
  WRITE(L,*) '          ENGLISH UNITS'
  WRITE(L,*) 'location degrees F      psi      lbm/sec'
  DO 20 I=1,51
    WRITE(L,32) I,(T(I)*1.8-460.),(P(I)*.145),
& (MDOT(I)*2.204585538)
20  CONTINUE
  WRITE(L,*) ' '
  WRITE(L,*) '          METRIC UNITS'
  WRITE(L,*) 'location degrees K      KPA      kg/sec'
  DO 21 I=1,51
    WRITE(L,32) I,T(I),P(I),MDOT(I)
21  CONTINUE
32  format(2x,i4,5x,f8.2,5x,f8.2,5x,f8.2)
  WRITE(L,*) ' '
  WRITE(L,*) '          COMBUSTOR 1 PRODUCTS'
  SSSUM=NN2+NO2+NCO2+NCO+NH2O+NH2+NCH4+NC2H6+NSO2+NH2S+NON+NNO
  WRITE(L,601) NN2/SSUM
  WRITE(L,602) NO2/SSUM
  WRITE(L,603) NCO2/SSUM
  WRITE(L,604) NCO/SSUM
  WRITE(L,605) NH2O/SSUM
  WRITE(L,606) NH2/SSUM
  WRITE(L,607) NCH4/SSUM
  WRITE(L,608) NC2H6/SSUM
  WRITE(L,609) NSO2/SSUM

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WRITE(L,610) NH2S/sssuum
WRITE(L,611) NOH/sssuum
WRITE(L,612) NNO/sssuum
WRITE(L,*) ' '
WRITE(L,*) '          COMBUSTOR 2 PRODUCTS'
sssuum=NH2S+NO2S+NCO2S+NCOS+NH2OS+NH2S+NCH4S+
+ NC2H6S+NSO2S+NH2SS+NOHS+NNOS
WRITE(L,601) NH2S/sssuum
WRITE(L,602) NO2S/sssuum
WRITE(L,603) NCO2S/sssuum
WRITE(L,604) NCOS/sssuum
WRITE(L,605) NH2OS/sssuum
WRITE(L,606) NH2S/sssuum
WRITE(L,607) NCH4S/sssuum
WRITE(L,608) NC2H6S/sssuum
WRITE(L,609) NSO2S/sssuum
WRITE(L,610) NH2SS/sssuum
WRITE(L,611) NOHS/sssuum
WRITE(L,612) NNOS/sssuum
601 FORMAT(27H          N2          -----> ,F11.5)
602 FORMAT(27H          O2          -----> ,F11.5)
603 FORMAT(27H          CO2         -----> ,F11.5)
604 FORMAT(27H          CO          -----> ,F11.5)
605 FORMAT(27H          H2O         -----> ,F11.5)
606 FORMAT(27H          H2          -----> ,F11.5)
607 FORMAT(27H          CH4         -----> ,F11.5)
608 FORMAT(27H          C2H6        -----> ,F11.5)
609 FORMAT(27H          SO2         -----> ,F11.5)
610 FORMAT(27H          H2S         -----> ,F11.5)
611 FORMAT(27H          OH          -----> ,F11.5)
612 FORMAT(27H          NO          -----> ,F11.5)
WRITE(L,*) ' '
WRITE(L,*) '          WORK INPUT/OUTPUT (KW)'
WRITE(L,*) ' '
WRITE(L,*) ' WORK FROM MAIN GAS TURBINE (GE MS7001 HEAVY DUTY)'
WRITE(L,621) WCOMP1
WRITE(L,622) WCOMP2
WRITE(L,623) WTURB1
WRITE(L,624) WTURB2
WRITE(L,*) ' '
WRITE(L,626) TOTWORK
WRITE(L,625) WTURBR
WRITE(L,627) (WTURBR+TOTWORK)
621 FORMAT(41H      Work of FIRST COMPRESSOR -----> ,F9.1)
622 FORMAT(41H      Work of SECOND COMPRESSOR -----> ,F9.1)
623 FORMAT(41H      Work of FIRST TURBINE -----> ,F9.1)
624 FORMAT(41H      Work of SECOND TURBINE (optional)-> ,F9.1)
625 FORMAT(41H      Net Work of RECOVERY TURBINES -----> ,F9.1)
626 FORMAT(46H      TOTAL OUTPUT OF GAS TURBINE (GE MS7001)--> ,F9.1)
627 FORMAT(46H      ***** TOTAL OUTPUT OF SYSTEM ***** -> ,F9.1)
IF(TEST.EQ.0) THEN
  WRITE(L,*) ' '
  WRITE(L,*) '          HIGHER HEATING VALUES'
  WRITE(L,*) '          KJ/KG          BTU/SCF'
  WRITE(L,631) HHVPPD,HHVPPD
  WRITE(L,632) HHVPPW,HHVPPW
631 FORMAT(2X,10HDY BASIS ,4X,F7.2,11X,F6.2)
632 FORMAT(2X,10HWET BASIS ,4X,F7.2,11X,F6.2)
  WRITE(L,*) ' '
  WRITE(L,*) '          LOWER HEATING VALUES'
  WRITE(L,*) '          KJ/KG          BTU/SCF'
  WRITE(L,633) LHVPPD,LHVPPD
  WRITE(L,634) LHVPPW,LHVPPW
633 FORMAT(2X,10KDRY BASIS ,4X,F7.2,11X,F6.2)
634 FORMAT(2X,10HWET BASIS ,4X,F7.2,11X,F6.2)

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```

WRITE(L,*) ' '
WRITE(L,*) ' HEAT RATES & EFFICIENCIES'
IF(TFACT.EQ.0) THEN
WRITE(L,*) ' '
WRITE(L,*) ' GAS TURBINE-THROTTLE AFTER GASIFIER'
WRITE(L,*) ' (HIGHER HEATING VALUES)'
WRITE(L,641) EF2COAL2
WRITE(L,642) HRC2
WRITE(L,643) EFGASW2
WRITE(L,644) HRGW2
WRITE(L,*) ' (LOWER HEATING VALUES)'
WRITE(L,641) EFFCOAL2
WRITE(L,642) HRCOAL2
WRITE(L,643) EFFGASW2
WRITE(L,644) HRGASW2
WRITE(L,*) ' '
WRITE(L,*) ' '
WRITE(L,*) 'GAS TURBINE-RECOVERY TURBINE-THROTTLE AFTER GASIFIER'
WRITE(L,*) ' (HIGHER HEATING VALUES)'
WRITE(L,641) EF2COAL4
WRITE(L,642) HRC4
WRITE(L,*) ' (LOWER HEATING VALUES)'
WRITE(L,641) EFFCOAL4
WRITE(L,642) HRCOAL4
WRITE(L,*) ' '
ENDIF
IF(TFACT.EQ.1) THEN
WRITE(L,*) ' '
WRITE(L,*) ' GAS TURBINE-TURBINE AFTER GASIFIER'
WRITE(L,*) ' (HIGHER HEATING VALUES)'
WRITE(L,641) EF2COAL1
WRITE(L,642) HRC1
WRITE(L,643) EFGASW1
WRITE(L,644) HRGW1
WRITE(L,*) ' (LOWER HEATING VALUES)'
WRITE(L,641) EFFCOAL1
WRITE(L,642) HRCOAL1
WRITE(L,643) EFFGASW1
WRITE(L,644) HRGASW1
WRITE(L,*) ' '
WRITE(L,*) ' '
WRITE(L,*) 'GAS TURBINE-RECOVERY TURBINE-TURBINE AFTER GASIFIER'
WRITE(L,*) ' (HIGHER HEATING VALUES)'
WRITE(L,641) EF2COAL3
WRITE(L,642) HRC3
WRITE(L,*) ' (LOWER HEATING VALUES)'
WRITE(L,641) EFFCOAL3
WRITE(L,642) HRCOAL3
WRITE(L,*) ' '
ENDIF
WRITE(L,*) ' RELATIVE FLOWS'
WRITE(L,645) (MDOT(21)/MDOT(20))
WRITE(L,646) ((MDOT(21)+MDOT(22))/MDOT(20))
WRITE(L,647) (MDOT(8)/MDOT(20))
WRITE(L,648) V1
WRITE(L,649) V2
WRITE(L,650) V3
641 FORMAT(30H COAL EFFICIENCY -----> ,F5.4)
642 FORMAT(30H COAL HEAT RATE -----> ,F7.1)
643 FORMAT(30H GAS EFFICIENCY -----> ,F5.4)
644 FORMAT(30H GAS HEAT RATE -----> ,F7.1)
645 FORMAT(30H LBM STEAM / LBM COAL ----> ,F7.4)
646 FORMAT(30H LBM STEAM&WATER / LBM COAL > ,F7.4)
647 FORMAT(30H LBM AIR / LBM COAL ---> ,F7.4)
648 FORMAT(F8.4,30H STD FT^3 GAS OUT / LBM COAL)

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649   FORMAT(F8.5,30H   LBM H2O / STD FT^3 GAS OUT )
650   FORMAT(F8.5,30H   STD FT^3 AIR / STD FT^3 GAS)
      ENDIF
      IF(TEST.EQ.1) THEN
        WRITE(L,*) ' '
        WRITE(L,*) ' (BASED ON LOWER HEATING VALUE)'
        WRITE(L,*) ' Efficiency ----> ',EFF
        WRITE(L,*) ' Heat Rate ----> ',HR
      ENDIF
797   CONTINUE
      CLOSE(13)
      END

C
      SUBROUTINE GASPROP(NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NN2,NH2S,NSO2,NO2
& ,CPPROD,MPROD,RPROD,CVPROD,KPROD,CP)
      DOUBLE PRECISION NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NN2,NH2S,NSO2,NO2,
& CPPROD,MPROD,RPROD,CVPROD,KPROD,CP(12)
      CPPROD=(NN2*CP(1)+NCO2*CP(3)+NCO*CP(4)+NH2O*CP(5)+NH2*CP(6)+
& NCH4*CP(7)+NC2H6*CP(8)+NSO2*CP(9)+NH2S*CP(10)+NO2*CP(2))/
& (NCO+NCO2+NH2+NH2O+NN2+NH2S+NCH4+NC2H6+NSO2+NO2)
      MPROD=(NCO*28.011+NCO2*44.01+NH2*2.016+NH2O*18.015+NN2*28.013+
& NH2S*34.0760+NCH4*16.043+NC2H6*30.070+NSO2*64.0588+NO2*
& 31.999)/(NCO+NCO2+NH2+NH2O+NN2+NH2S+NCH4+NC2H6+NSO2+NO2)
      CPPROD=CPPROD/MPROD
      RPROD=8.3144/MPROD
      CVPROD=CPPROD-RPROD
      KPROD=CPPROD/CVPROD
      RETURN
      END

C
      SUBROUTINE GASIFIER(MC,MH2,MO2,MN2,MS,PERH2O,PERASH,HHV,NN2,NO2,
& NCO2,NCO,NH2O,NH2,NCH4,NC2H6,NSO2,NH2S,NOH,MNO,NLIGH2O,TLIQ,
& HSTEAM,TSTEAM,GASEFF,TPROD,Z1,Z2,TAIR,MASH,MH2O)
      DOUBLE PRECISION NCO2,NCO,NH2,MNO,
& A(11),B(11),T(2),X(5),H(11),S(11),G(11),ECONST(5),
& DIFF(3),NCH4,NC2H6,NSO2,NH2S,NC,NS,Z1,Z2,ID(11),H298(11),
& S298(11),M(11),NLIGH2O,NAIR,TLIQ,TSTEAM,TAIR,NC1,NH21,NO21,
& NS1,SUMN,HSTEAM,NN21,MC,MH2,MO2,MN2,MS,NH2O,NH2,NO2,HHV,
& DIF1,DIF2,DIF3,DIF4,DIF5,DIF6,MCOAL,MASH,PERASH,
& PERH2O,MH2O,MEXTRAS,PEREXTRAS,NFACT,
& GASEFF,NOH,TPROD,DIF7,SUM1,P
      INTEGER I,J,TEST,COUNT
      OPEN(UNIT=11,FILE='DATA1',STATUS='UNKNOWN')
      DO 10 J=1,10
        READ(11,*) ID(J),A(J),B(J),H298(J),S298(J),M(J)
10    CONTINUE
      CLOSE(11)
      T(1)=1000.
      T(2)=860.
      NFACT=1
      COUNT=0.
C *****
C ** MAIN LOOP **
C *****
      P=20.*101.325
      NC=MC/12.01
      NH2=MH2/2.016
      NO2=NO2/32.000
      NN2=NN2/28.013
      NS=MS/32.06
      SUMN=NC+NH2+NO2+NN2+NS
      SUM1=SUMN
      NC=NC/SUMN
      NC1=NC
      NH2=NH2/SUMN

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NH21=NH2
NO2=NO2/SUMN
NO21=NO2
NN2=NN2/SUMN
NN21=NN2
NS=NS/SUMN
NS1=NS
NAIR=Z2
NLIQH2O=.36d0*Z1
NSTEAM=(1.0d0-.36d0)*Z1
MCOAL=NC*12.011+NH2*2.016+NO2*31.999+NN2*28.013+NS*32.064
PEREXTRAS=PERH2O+PERASH
IF(PEREXTRAS.EQ.0) GO TO 25
MEXTRAS=(PEREXTRAS*MCOAL)/(1.0d0-PEREXTRAS)
MASH=(PERASH/PEREXTRAS)*MEXTRAS
MH2O=(PERH2O/PEREXTRAS)*MEXTRAS
Z1=Z1+MH2O/18.
25 CONTINUE
20 COUNT=COUNT+1.
DO 40 I=1,2
DO 30 J=1,12
H(J)=H298(J)+A(J)*(T(I)-298.)-B(J)*DLOG(T(I)/298.)
S(J)=S298(J)+A(J)*DLOG(T(I)/298.)+B(J)*(1./T(I))-1./298.)
G(J)=H(J)-T(I)*S(J)
30 CONTINUE
ECONST(1)=((G(6)+G(3)-G(5)-G(4))/(8.31434*T(I)))
ECONST(2)=((G(4)+3*G(6)-G(7)-G(5))/(8.31434*T(I)))
ECONST(3)=((2*G(4)+5*G(6)-G(8)-2.*G(5))/(8.31434*T(I)))
ECONST(4)=((G(10)+2*G(5)-G(9)-3*G(6))/(8.31434*T(I)))
CALL EQUIL(ECONST,P,X,Z1,Z2,NC1,NH21,NO21,NN21,NS1,T(I),NFACT)
NCO=NC1-2.*Z2-2.*NO21+X(1)-X(2)-2.*X(3)
NCO2=2*Z2+2*NO21-X(1)
NH2=NH21+NC1-NS1-X(1)-3.*X(2)-5.*X(3)+3.*X(4)
NH2O=Z1-NC1+X(1)+X(2)+2.*X(3)-2.*X(4)
NN2=NFACT*3.77*Z2+NN21
NH2S=NS1-X(4)
NCH4=X(2)
NC2H6=X(3)
NSO2=X(4)
NO2=0.D0
DIF1=(1.-GASEFF)*MCOAL*MHV
DIF2=(NLIQH2O+MH2O/18.)*(-285838.+(72.8815*(TLIQ-25)+
& .026086*(TLIQ**2.-625.)))
& DIF3=NSTEAM*(-241827+42.02*(TSTEAM-298.)-
& 3050.7*DLOG(TSTEAM/298.))
& DIF4=(NC1*(-393522)+NH21*(-285838)+NS1*(-296847)+
& MHV*(NC1*12.011+NH21*2.016+NO21*31.999+NN21*28.013+
& NS1*32.06))
& DIF5=(NCO*H(4)+NCO2*H(3)+NH2*H(6)+NH2O*H(5)+NN2*H(1)+
& NH2S*H(10)+NCH4*H(7)+NC2H6*H(8)+NSO2*H(9))
DIF6=-MASH*1.100*(1000-298.15)
DIF7=NAIR*(0.+A(2)*(TAIR-298.)-B(2)*DLOG(TAIR/298.))+
& 3.77*NAIR*(0.+A(1)*(TAIR-298.)-B(1)*DLOG(TAIR/298.))
DIF(1)=DIF1+DIF2+DIF3+DIF4+DIF5+DIF6+DIF7
40 CONTINUE
T(2)=(T(2)*DIF(1)-T(1)*DIF(2))/(DIF(1)-DIF(2))
TEST=ABS(DIF(2))
IF(TEST.GT..1) GO TO 20
TPROD=T(2)
RETURN
END
C
SUBROUTINE EQUIL(ECONST,P,X,Z1,Z2,NC1,NH21,NO21,NN21,NS1,T,NFACT)
DOUBLE PRECISION ECONST(4),X(4),A(4,4),B(4),F(4),FO(4),
& DX(4),Z1,Z2,NC1,NH21,NO21,NN21,NS1,T,NFACT,ACON,BCON,

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```

& NH22,NH202,P,DELX
INTEGER J,TEST,I
IF(T.GT.950.) THEN
  ACON=2.77528D-1
  BCON=-9.97464D-1
  X(1)=ACON*DEXP(-ECONST(1)*BCON)
  ACON=3.27707D-1
  BCON=-2.18632D-1
  X(2)=ACON*DEXP(-ECONST(2)*BCON)
  ACON=4.47123D-4
  BCON=-1.99404D-1
  X(3)=ACON*DEXP(-ECONST(3)*BCON)
  ACON=1.18176D-7
  BCON=-4.24760D-1
  X(4)=ACON*DEXP(-ECONST(4)*BCON)
ENDIF
IF(T.LE.950) THEN
  ACON=2.16083D-1
  BCON=-6.72486D-1
  X(1)=ACON*DEXP(-ECONST(1)*BCON)
  ACON=2.41669D-1
  BCON=4.10778D-2
  X(2)=ACON*DEXP(-ECONST(2)*BCON)
  ACON=3.41324D-5
  BCON=8.61858D-3
  X(3)=ACON*DEXP(-ECONST(3)*BCON)
  ACON=5.56312D-8
  BCON=-3.80523D-1
  X(4)=ACON*DEXP(-ECONST(4)*BCON)
ENDIF
10 CALL KSUB(X,P,FO,Z1,Z2,NC1,NH21,NO21,NN21,NS1,NFACT)
DO 30 I=1,4
  DELX=.01*X(I)
  X(I)=X(I)+DELX
  CALL KSUB(X,P,F,Z1,Z2,NC1,NH21,NO21,NN21,NS1,NFACT)
  DO 20 J=1,4
    A(J,I)=(F(J)-FO(J))/DELX
20 CONTINUE
  X(I)=X(I)-DELX
  B(I)=ECONST(I)-FO(I)
30 CONTINUE
CALL GAUSS(A,DX,B,4)
TEST=0.
DO 40 I=1,4
  X(I)=X(I)+DX(I)
  TEST=TEST+ABS(DX(I)/X(I))
40 CONTINUE
NH22=NH21+NC1-NS1-X(1)-3.*X(2)-5.*X(3)+3.*X(4)
NH202=Z1-NC1+X(1)+X(2)+2.*X(3)-2.*X(4)
IF(NH22.LT.0) THEN
  X(2)=(-.01+NH22+3.*X(2))/3.
ENDIF
IF(NH202.LT.0) THEN
  X(1)=.01-Z1-NC1-X(2)-2.*X(3)+2.*X(4)
ENDIF
IF(TEST.GT.0.00001) GO TO 10
RETURN
END

```

C

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SUBROUTINE KSUB(X,P,F,Z1,Z2,NC1,NH21,NO21,NN21,NS1,NFACT)
DOUBLE PRECISION X(4),F(4),Z1,Z2,NC1,NH21,NO21,NN21,NS1,NFACT,
& YCO,YCO2,YH2,YH2O,YH2S,YCH4,YC2H6,YSO2,SUMM,P
SUMM=NC1-2.*Z2-2.*NO21+X(1)-X(2)-2.*X(3)+2.*Z2+2.*NO21-X(1)+NH21+
& NC1-NS1-X(1)-3.*X(2)-5.*X(3)+3.*X(4)+Z1-NC1+X(1)+X(2)+2.*X(3)-
& 2.*X(4)+NFACT*3.77*Z2+NN21+NS1-X(4)+X(2)+X(3)+X(4)

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YCO=(NC1-2.*Z2-2.*NO21+X(1)-X(2)-2.*X(3))/SUMM
YCO2=(2.*Z2+2.*NO21-X(1))/SUMM
YH2=(NH21+NC1-NS1-X(1)-3.*X(2)-5.*X(3)+3.*X(4))/SUMM
YH2O=(Z1-NC1+X(1)+X(2)+2.*X(3)-2.*X(4))/SUMM
YH2=(NFACT*3.77*Z2+NH21)/SUMM
YH2S=(NS1-X(4))/SUMM
YCH4=X(2)/SUMM
YC2H6=X(3)/SUMM
YSO2=X(4)/SUMM
F(1)=DLOG(((YH2O**P/100.)*(YCO**P/100.))/
& ((YH2**P/100.)*(YCO2**P/100.)))
F(2)=DLOG(((YCH4**P/100.)*(YH2O**P/100.))/
& ((YCO**P/100.)*(YH2**P/100.))**3.))
F(3)=DLOG(((YC2H6**P/100.)*(YH2O**P/100.))**2.)/
& ((YCO**P/100.))**2.*(YH2**P/100.))**5.))
F(4)=DLOG(((YSO2**P/100.)*(YH2**P/100.))**3.)/
& ((YH2S**P/100.)*(YH2O**P/100.))**2.))
RETURN
END

```

C

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SUBROUTINE COMBUSTOR(NN2,Z, NCO2,NCO,NH2O,NH2,NCH4,NC2H6,
& NSO2,NH2S,TAIR,TREACT,TPROD,P,N,MO2,HHV,COMEFF)
DOUBLE PRECISION NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NN2,NH2S,NSO2,
& MO2,NNO,NOH,A(12),B(12),T(2),X(7),H(12),
& S(12),G(12),ECONST(12),DIFF(2),ID(12),H298(12),S298(12),
& M(12),TAIR,TREACT,TPROD,NCO21,NCO1,NH21,NCH41,NC2H61,
& NH2O1,NH21,NH2S1,NSO21,NO21,NNO1,NOH1,DIF1,DIF2,DIF3,DIF4,
& NFACT,P,HR(12),HA(12),TR,TA,Z,COMEFF,HHV,MPROD
INTEGER N,TEST,I,COUNT,J
OPEN(UNIT=12,FILE='DATA1',STATUS='UNKNOWN')
DO 10 J=1,12
  READ(12,*) ID(J),A(J),B(J),H298(J),S298(J),M(J)
10 CONTINUE
CLOSE(12)
NN21=NN2
NCO21=NCO2
NCO1=NCO
NH2O1=NH2O
NH21=NH2
NCH41=NCH4
NC2H61=NC2H6
NSO21=NSO2
NH2S1=NH2S
NFACT=1.0d0
DO 12 J=1,12
  TR=TREACT
  TA=TAIR
  HR(J)=H298(J)+A(J)*(TR-298.)-B(J)*DLOG(TR/298.)
  HA(J)=H298(J)+A(J)*(TA-298.)-B(J)*DLOG(TA/298.)
12 CONTINUE
T(1)=1300.
T(2)=1350.
MPROD=(NCO*28.011+NCO2*44.01+NH2*2.016+NH2O*18.015+NN2*28.013+
& NH2S*34.0760+NCH4*16.043+NC2H6*30.070+NSO2*64.0588+NO2*
& 31.999)/(NCO+NCO2+NH2+NH2O+NN2+NH2S+NCH4+NC2H6+NSO2+NO2)
HHV=HHV*MPROD
COUNT=0.
C *****
C ** MAIN LOOP **
C *****
20 COUNT=COUNT+1.
DO 40 I=1,2
  DO 30 J=1,12
    H(J)=H298(J)+A(J)*(T(I)-298.)-B(J)*DLOG(T(I)/298.)
    S(J)=S298(J)+A(J)*DLOG(T(I)/298.)+B(J)*(1./T(I)-1./298.)

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      G(J)=H(J)-T(1)*S(J)
30  CONTINUE
      ECONST(1)=(2.*G(3)-2.*G(4)-G(2))/(8.31434*T(1))
      ECONST(2)=(2.*G(9)+2.*G(5)-2.*G(10)-3.*G(2))/(8.31434*T(1))
      ECONST(3)=(2.*G(5)-2.*G(6)-G(2))/(8.31434*T(1))
      ECONST(4)=(2.*G(5)-G(6)-2.*G(11))/(8.31434*T(1))
      ECONST(5)=(G(1)+G(2)-2.*G(12))/(8.31434*T(1))
      ECONST(6)=(G(3)+2.*G(5)-2.*G(2)-G(7))/(8.31434*T(1))
      ECONST(7)=(2.*G(3)+3.*G(5)-3.5*G(2)-G(8))/(8.31434*T(1))
      X(1)=0.
      X(2)=0.
      X(3)=0.
      X(4)=0.
      X(5)=0.
      X(6)=0.
      X(7)=0.
      NN2=Z*3.77*HFACT+NN21-X(5)
      NO2=Z-.5*NCO1-.5*NH21-2.*NCH41-3.5*NC2H61-1.5*NH2S1+X(1)+
&      3.*X(2)+X(3)-X(5)+2.*X(6)+3.5*X(7)
      NCO2=NCO21+NCO1+NCH41+2.*NC2H61-2.*X(1)-X(6)-2.*X(7)
      NCO=2.*X(1)
      NH2O=NH21+NH2O1+2.*NCH41+3.*NC2H61+NH2S1-2.*X(2)-2.*X(3)-
&      2.*X(4)-2.*X(6)-3.*X(7)
      NH2=2.*X(3)+X(4)
      NCH4=X(6)
      NC2H6=X(7)
      NSO2=NH2S1+NSO21-2.*X(2)
      NH2S=2.*X(2)
      NOH=2.*X(4)
      NNO=2.*X(5)
      DIF1=(COMEFF-1.)*HHV
      DIF2=NCO21*HR(3)+NCO1*HR(4)+NH21*HR(6)+NCH41*HR(7)+
&      NC2H61*HR(8)+NH2O1*HR(5)+NN21*HR(1)+NH2S1*HR(10)+
&      NSO21*HR(9)+NO21*HR(2)+NNO1*HR(12)+NOH1*HR(11)
      DIF3=Z*3.77*HA(1)+Z*HA(2)
      DIF4=- (NCO*H(4)+NCO2*H(3)+NH2*H(6)+NH2O*H(5)+NN2*H(1)+
&      NH2S*H(10)+NCH4*H(7)+NC2H6*H(8)+NSO2*H(9)+NNO*H(12)+
&      NOH*H(11)+NO2*H(2))
      DIFF(1)=DIF1+DIF2+DIF3+DIF4
40  CONTINUE
      T(2)=(T(2)*DIFF(1)-T(1)*DIFF(2))/(DIFF(1)-DIFF(2))
      T(1)=.95*T(2)
      TEST=ABS(DIFF(2))
      IF(TEST.GT..1) GO TO 20
50  TPROD=T(2)
      RETURN
      END
C
      SUBROUTINE COMBUSTOR2(NN2,NO2,NCO2,NCO,NH2O,NH2,NCH4,NC2H6,
&      NSO2,NH2S,NNO,NOH,MCF,MH2F,MO2F,MN2F,MSF,HHVC,MASH,TREACT,
&      TPROD,P,COMEFF,MFINES,DIFF)
      DOUBLE PRECISION NCO2,NCO,NH2,NCH4,NC2H6,NH2O,NN2,NH2S,NSO2,
&      NO2,NNO,NOH,A(12),B(12),X(7),H(12),MFINES,
&      S(12),G(12),ECONST(12),DIFF,ID(12),H298(12),S298(12),
&      M(12),TAIR,TREACT,TPROD,NCO21,NCO1,NH21,NCH41,NC2H61,
&      NH2O1,NH21,NH2S1,NSO21,NO21,NNO1,NOH1,DIF1,DIF2,DIF3,DIF4,
&      P,HR(12),TR,COMEFF,HHVC,MPROD,TEMP,MC,MH2,MO2,MN2,MH2O1,
&      MS,MC1,MH21,MO21,MN21,MS1,MASH,MCF,MH2F,MO2F,MN2F,MSF
      INTEGER TEST,I,J
      OPEN(UNIT=12,FILE='DATA1',STATUS='UNKNOWN')
      DO 10 J=1,12
          READ(12,*) ID(J),A(J),B(J),H298(J),S298(J),M(J)
10  CONTINUE
      CLOSE(12)
      TEMP=949.8167

```

```

NN21=NN2
NO21=NO2
NCO21=NCO2
NCO1=NCO
NH201=NH20
NH21=NH2
NCH41=NCH4
NC2H61=NC2H6
NSO21=NSO2
NH2S1=NH2S
NNO1=NNO
NOH1=NOH
MC1=MC1
MH21=MH2F
MO21=MO2F
MN21=MN2F
MS1=MSF

```

C

```

DO 12 J=1,12
  TR=TREACT
  HR(J)=H298(J)+A(J)*(TR-298.)-B(J)*DLOG(TR/298.)
  H(J)=H298(J)+A(J)*(TEMP-298.)-B(J)*DLOG(TEMP/298.)
  G(J)=H(J)-TEMP*S(J)

```

12

CONTINUE

C

```

ECONST(1)=(2.*G(3)-2.*G(4)-G(2))/(8.31434*TEMP)
ECONST(2)=(2.*G(9)+2.*G(5)-2.*G(10)-3.*G(2))/(8.31434*TEMP)
ECONST(3)=(2.*G(5)-2.*G(6)-G(2))/(8.31434*TEMP)
ECONST(4)=(2.*G(5)-G(6)-2.*G(11))/(8.31434*TEMP)
ECONST(5)=(G(1)+G(2)-2.*G(12))/(8.31434*TEMP)
ECONST(6)=(G(3)+2.*G(5)-2.*G(2)-G(7))/(8.31434*TEMP)
ECONST(7)=(2.*G(3)+3.*G(5)-3.5*G(2)-G(8))/(8.31434*TEMP)
X(1)=0.
X(2)=0.
X(3)=0.
X(4)=0.
X(5)=0.
X(6)=0.
X(7)=0.
NN2=NN21+MN21-X(5)
NO2=NO21+MO21-.5*NCO1-2.*NCH41-3.5*NC2H61-1.5*NH2S1-.5*NH21-
& MC1-.5*MH21-MS1+X(1)+3.*X(2)+X(3)-X(5)+2.*X(6)+3.5*X(7)
NCO2=NCO21+NCO1+NCH41+2.*NC2H61+MC1-2.*X(1)-X(6)-2.*X(7)
NCO=2.*X(1)
NH20=NH21+NH201+2.*NCH41+3.*NC2H61+NH2S1+MH21+MH201-2.*X(2)
& -2.*X(3)-2.*X(4)-2.*X(6)-3.*X(7)
NH2=2.*X(3)+X(4)
NCH4=X(6)
NC2H6=X(7)
NSO2=NH2S1+NSO21+MS1-2.*X(2)
NH2S=2.*X(2)
NOH=2.*X(4)
NNO=2.*X(5)
DIF1=(COMEFF-1.)*MFINES*HHVC
DIF2=NCO21*HR(3)+NCO1*HR(4)+NH21*HR(6)+NCH41*HR(7)+
& NC2H61*HR(8)+NH201*HR(5)+NN21*HR(1)+NH2S1*HR(10)+
& NSO21*HR(9)+NO21*HR(2)+NNO1*HR(12)+NOH1*HR(11)
DIF3=-(NCO*H(4)+NCO2*H(3)+NH2*H(6)+NH20*H(5)+NN2*H(1)+
& NH2S*H(10)+NCH4*H(7)+NC2H6*H(8)+NSO2*H(9)+NNO*H(12)+
& NOH*H(11)+NO2*H(2))
DIF4=-MASH*1.100*(949.8-298.15)
DIFF=DIF1+DIF2+DIF3+DIF4
TPROD=949.8
RETURN
END

```



```

C
SUBROUTINE GAUSS(A,X,B,N)
DOUBLE PRECISION A(N,N),X(N),B(N),T
INTEGER I,M,J,NM,N,KP,K
NM=N-1
DO 20 K=1,NM
  KP=K+1
  DO 20 I=KP,N
    T=A(I,K)/A(K,K)
    DO 10 J=K,N
10   A(I,J)=A(I,J)-T*B(K)
20   B(I)=B(I)-T*B(K)
    X(N)=B(N)/A(N,N)
    K=N
    DO 40 M=1,NM
      K=K-1
      KP=K+1
      X(K)=B(K)
      DO 30 J=KP,N
30     X(K)=X(K)-A(K,J)*X(J)
40     X(K)=X(K)/A(K,K)
    RETURN
END

```

```

C
SUBROUTINE GASTEMP(MC,MH2,MO2,MN2,MS,PERH2O,PERASH,HHV,MN2,NO2,
& NCO2,NCO,MH2O,MH2,NCH4,NC2H6,NSO2,NH2S,NOH,NNO,NLIQH2O,TLIQ,
& NSTEAM,TSTEAM,GASEFF,TPROD,Z1,Z2,TAIR,MASH,MH2O)
DOUBLE PRECISION MC,MH2,MO2,MN2,MS,PERH2O,PERASH,HHV,MN2,NO2,
& NCO2,NCO,MH2O,MH2,NCH4,NC2H6,NSO2,NH2S,NOH,NNO,NLIQH2O,TLIQ,
& NSTEAM,TSTEAM,T(?),MCOAL,PEREXTRAS,MEXTRAS,MASH,MH2O,GASEFF,
& NC1,MH21,NO21,MN21,NS1,DIF1,DIF2,DIF3,DIF4,DIF5,DIF6,DIFF(2),
& TEST,TPROD,AMOUNTC,RATIO,DIF7,ID(12),A(12),B(12),H298(12),
& S298(12),M(12),H(12),TAIR,Z1,Z2,NAIR,SUMM
INTEGER COUNT,I,J
OPEN(UNIT=11,FILE='DATA1',STATUS='UNKNOWN')
DO 10 J=1,10
  READ(11,*) ID(J),A(J),B(J),H298(J),S298(J),M(J)
10 CONTINUE
CLOSE(11)
T(1)=800.
T(2)=900.
NC1=MC/12.01
MH21=MH2/2.016
NO21=NO2/32.000
MN21=MN2/28.013
NS1=MS/32.06
SUMM=NC1+MH21+NO21+MN21+NS1
NC1=NC1/SUMM
MH21=MH21/SUMM
NO21=NO21/SUMM
MN21=MN21/SUMM
NS1=NS1/SUMM
AMOUNTC=NCO2+NCO+NCH4+2.*NC2H6
RATIO=NC1/AMOUNTC
MH2=MH2*RATIO
NCO2=NCO2*RATIO
NCO=NCO*RATIO
MH2=MH2*RATIO
NCH4=NCH4*RATIO
NC2H6=NC2H6*RATIO
NH2S=NS1
Z2=(MN2-MN21)/3.77
NAIR=Z2
NLIQH2O=.36*Z1
NSTEAM=(1.-.36)*Z1

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MCOAL=NC1*12.011+NH21*2.016+NO21*31.999+NN21*28.013+NS1*32.664
PEREXTRAS=PERH2O+PERASH
IF(PEREXTRAS.EQ.0) GO TO 25
MEXTRAS=(PEREXTRAS*MCOAL)/(1.-PEREXTRAS)
MASH=(PERASH/PEREXTRAS)*MEXTRAS
MH2O=(PERH2O/PEREXTRAS)*MEXTRAS
Z1=Z1+MH2O/18.
NH2O=NH21+Z1-NH2-2.*NCH4-3.*NC2H6-NS1
25 CONTINUE
20 COUNT=COUNT+1.
DO 40 I=1,2
DO 30 J=1,2
H(J)=H298(J)+A(J)*(T(I)-298.)-B(J)*DLOG(T(I)/298.)
30 CONTINUE
DIF1=-(1.-GASEFF)*MCOAL*HHV
DIF2=(NLIQH2O+MH2O/18.)*(-285838.+(72.8815*(TLIQ-25)+
& .026086*(TLIQ**2.-625.)))
& DIF3=NSTEAM*(-241827+42.02*(TSTEAM-298.)-
& 3050.7*DLOG(TSTEAM/298.))
DIF4=(NC1*(-393522)+NH21*(-285838)+NS1*(-296847)+
& HHV*(NC1*12.011+NH21*2.016+NO21*31.999+NN21*28.013+
& NS1*32.664))
DIF5=-(NCO*H(4)+NCO2*H(3)+NH2*H(6)+NH2O*H(5)+NN2*H(1)+
& NH2S*H(10)+NCH4*H(7)+NC2H6*H(8)+NSO2*H(9))
DIF6=-MASH*1.100*(1000-298.15)
DIF7=MAIR*(0.+A(2)*(TAIR-298.)-B(2)*DLOG(TAIR/298.))+
& 3.77*MAIR*(0.+A(1)*(TAIR-298.)-B(1)*DLOG(TAIR/298.))
DIFF(1)=DIF1+DIF2+DIF3+DIF4+DIF5+DIF6+DIF7
40 CONTINUE
T(2)=(T(2)*DIFF(1)-T(1)*DIFF(2))/(DIFF(1)-DIFF(2))
T(1)=1.05*T(2)
TEST=ABS(DIFF(2))
IF(TEST.GT..1) GO TO 20
TPROD=T(2)
RETURN
END

```

C

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SUBROUTINE HEATRECOVERY(T,P,MDOT,WNET,WPUMPR1,WPUMPR2,WPUMPR3,
& WTURBR1,WTURBR2,WTURBR3,WTURBR4,QSUPER,QHPBOIL,QLPBOIL,
& QCONDENSER,QPIPELOSS,CPPROD)
DOUBLE PRECISION WNET,TA1,TA2,TA3,TA4,P1,P2,P3,P4,P5,P6,P7,P8,P9,
& P10,P11,P12,P13,P14,P15,P16,
& CPAIR,MDOTAIR,MP,MT,
& MDOTGAS,CPWATER,CPWATER1,CPWATER2,UABOIL1,UABOIL2,WTURBR1,UA1,
& WTURBR2,WTURBR3,WTURBR4,WTURB,WPLWP,WPUMP1,WPUMP2,WPUMP3,C1,C2,
& DIFFA3,DIFFA4,RCRIT1,RCRIT2,RCRIT3,RCRIT4,R1,R2,R3,R4,MDOTB1,
& MDOTB2,MDOTF,
& COUNTA2,COUNTA3,COUNTA4,VF1,VG1,HF1,HG1,SF1,SG1,VF3,VG3,HF3,
& HG3,SF3,SG3,VF5,VG5,HF5,HG5,SF5,SG5,UA2,RSTAR1,RSTAR2,V6,H6,S6,
& V12,H12,S12,VF12,VG12,HF12,HG12,SF12,SG12,V10,H10,S10,VF10,
& VG10,HF10,HG10,SF10,SG10,X10,V15,H16,S16,VF16,VG16,HF16,HG16,
& SF16,SG16,H2,H4,V7,H7,S7,T12T,T16T,H8,S8,H9,S9,H11,S11,
& TESTA2,TESTA3,TESTA4,DIFFA2,UASUPER,DTMEAN,TA2PRIME,VF15,VG15,
& HF15,HG15,SF15,SG15,H13,S13,H14,QINH,T1,T2,T3,T4,T5,T6,T7,T8,T9,
& T10,T11,T12,T13,T14,T15,T16,MDOTGASE,QPIPELOSS,MDOT,T,P,QREJ,
& CPPROD,QCONDENSER,QSUPER,QHPBOIL,QLPBOIL,WTURB1,WTURB2,WTURB3,
& WTURB4,WPUMPR1,WPUMPR2,WPUMPR3,QGAS,QBD
INTEGER I
DIMENSION C1(2),DIFFA3(2),C2(2),DIFFA4(2),MDOT(60),T(60),P(60)
P1=10.000
P2=140.000
P3=140.000
P4=6276.000
P5=P4
P6=P4

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P14=1207.000
P7=.9500*P6
P8=2172.000
P9=P14*.9500
P10=P14*.9500
P11=140.000/.9500
P12=140.000
P13=10.000
P15=P14
P16=P14*.9500
TA1=T(19)
T6=838.7000
T7=T6-3.000
NP=.600
NT=.7500
UABOIL1=1082.700
UABOIL2=700.000
MDOTAIR=.98500*MDOT(19)
MDOTGAS=MDOT(22)
CPAIR=CPPROD
CPWATER=4.16700
CALL DSAT(T1,P1,VF1,VG1,HF1,HG1,SF1,SG1,2)
WPUMP1=VF1*(P2-P1)/NP
T2=WPUMP1/CPWATER+T1
H2=WPUMP1+HF1
CALL DSAT(T3,P3,VF3,VG3,HF3,HG3,SF3,SG3,2)
WPUMP2=VF3*(P4-P3)/NP
T4=WPUMP2/CPWATER+T3
H4=WPUMP2+HF3
WPUMP3=VF3*(P14-P3)/NP
T14=WPUMP3/CPWATER+T3
H14=WPUMP3+HF3
TA2=480.000+273.1500
CALL DSAT(T5,P5,VF5,VG5,HF5,HG5,SF5,SG5,2)
CALL DSUPER(T6,P6,V6,H6,S6,1)
COUNTA2=0.000
10 COUNTA2=COUNTA2+1.000
COUNTA3=0.000
CPWATER1=(HF5-H4)/(T5-T4)
RCRIT1=(HG5-HF5)/CPWATER1/(TA2-T5)
RCRIT2=CPWATER1*(TA2-T4)/(HG5-HF5+CPWATER1*(T5-T4))
IF(RCRIT1.GE.RCRIT2)THEN
  RSTAR1=RCRIT1
ELSE
  RSTAR1=RCRIT2
ENDIF
R1=RSTAR1/(1.00-DEXP(-UABOIL1/MDOTAIR/CPAIR))
R2=R1+(UABOIL1/MDOTAIR/CPAIR)*(R1-RSTAR1)
C1(1)=1/R1
C1(2)=1/R2
20 COUNTA3=COUNTA3+1.000
DO 30 I=1,2
  UA1=MDOTAIR*CPAIR*(C1(I)/(C1(I)-1.000)*DLOG((TA2-T5-C1(I))*(HG5-
& HF5)/CPWATER1)/(TA2-T4-C1(I))*((HG5-HF5)/CPWATER1+T5-T4)))+
& DLOG((TA2-T5)/(TA2-T5-C1(I))*((HG5-HF5)/CPWATER1)))
30 DIFFA3(I)=UA1-UABOIL1
TESTA3=DABS(DIFFA3(2))
C1(2)=(C1(2)*DIFFA3(1)-C1(1)*DIFFA3(2))/(DIFFA3(1)-DIFFA3(2))
C1(1)=.97*C1(2)
IF(COUNTA3.GT.20.000)GO TO 99
IF(TESTA3-.1)40,40,20
40 MDOTB1=C1(2)*CPAIR/CPWATER1*MDOTAIR
TA3=TA2-C1(2)*(T5-T4+(HG5-HF5)/CPWATER1)
TA2PRIME=TA1-.9700*MDOTB1*(H6-HG5)/CPAIR/MDOTAIR
DIFFA2=TA2-TA2PRIME

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TESTA2=DABS(DIFFA2)
TA2=(TA2+TA2PRIME)/2.000
IF(COUNTA2.GT.90.000)GO TO 99
IF(TESTA2-.100)50,50,10
50 DTMEAN=(TA1-T6-TA2+T5)/DLOG((TA1-T6)/(TA2-T5))
UASUPER=MDOTAIR*CPAIR*(TA1-TA2)/DTMEAN
COUNTA4=0.000
CALL DSAT(T15,P15,VF15,VG15,HF15,HG15,SF15,SG15,2)
CPWATER2=(HF15-H14)/(T15-T14)
RCRIT3=(HG15-HF15)/CPWATER2/(TA3-T15)
RCRIT4=CPWATER2*(TA3-T14)/(HG15-HF15+CPWATER2*(T15-T14))
IF(RCRIT3.GE.RCRIT4)THEN
  RSTAR2=RCRIT3
ELSE
  RSTAR2=RCRIT4
ENDIF
R3=RSTAR2/(1.000-DEXP(-UABOIL2/CPAIR/MDOTAIR))
R4=R3+(UABOIL2/MDOTAIR/CPAIR)*(R1-RSTAR2)
C2(1)=1/R3
C2(2)=1/R4
60 COUNTA4=COUNTA4+1.000
DO 70 I=1,2
  UA2=MDOTAIR*CPAIR*(C2(I)/(C2(I)-1.000)*DLOG((TA3-T15-C2(I)*(HG15
& -HF15)/CPWATER2)/(TA3-T14-C2(I)*(HG15-HF15)/CPWATER2+T15-
& T14)))+DLOG((TA3-T15)/(TA3-T15-C2(I)*(HG15-HF15)/
& CPWATER2)))
70 DIFFA4(I)=UA2-UABOIL2
TESTA4=DABS(DIFFA4(I))
C2(2)=(C2(2)*DIFFA4(1)-C2(1)*DIFFA4(2))/(DIFFA4(1)-DIFFA4(2))
C2(1)=.9700*C2(2)
IF(COUNTA4.GT.90.00)GO TO 98
IF(TESTA4-.1)80,80,60
80 MDOTB2=C2(2)*CPAIR/CPWATER2*MDOTAIR
TA4=TA3-C2(2)*(T15-T14+(HG15-HF15)/CPWATER2)
CALL DSUPER(T7,P7,V7,H7,S7,1)
CALL DTURBINE(NT,P7,P8,T7,T8,H7,H8,S7,S8,WTURB1)
CALL DTURBINE(NT,P8,P9,T8,T9,H8,H9,S8,S9,WTURB2)
CALL DSAT(T16T,P16,VF16,VG16,HF16,HG16,SF16,SG16,2)
T16=T15-3.000
IF(T16.GT.T16T)THEN
  CALL DSUPER(T16,P16,V16,H16,S16,1)
ELSE
  CALL DSAT(T16,P16,VF16,VG16,HF16,HG16,SF16,SG16,2)
  H16=HG16
  S16=SG16
ENDIF
H10=(( .9700*MDOTB1-MDOTGAS)*H9+.9700*MDOTB2*H16)/(.9700*MDOTB1+
& .9700*MDOTB2-MDOTGAS)
CALL DSAT(T10,P10,VF10,VG10,HF10,HG10,SF10,SG10,2)
IF(H10.GT.HG10)THEN
  CALL DSUPER(T10,P10,V10,H10,S10,3)
ELSE
  X10=(H10-HF10)/(HG10-HF10)
  S10=X10*SG10+(1.000-X10)*SF10
ENDIF
CALL DTURBINE(NT,P10,P11,T10,T11,H10,H11,S10,S11,WTURB3)
CALL DTURBINE(NT,P11,P13,T11,T13,H11,H13,S11,S13,WTURB4)
CALL DSAT(T12T,P12,VF12,VG12,HF12,HG12,SF12,SG12,2)
IF(T12.GT.T12T)THEN
  CALL DSUPER(T12,P12,V12,H12,S12,1)
ELSE
  H12=HG12
  S12=SG12
  T12=T12T
ENDIF

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```

MDOTF=(MDOTB1+MDOTB2)*(HF3-H2)/(H12-H2)
WTURB=.97D0*MDOTB1*WTURB1+(.97D0*MDOTB1-MDOTGAS)*WTURB2+(.97D0*
& MDOTB1+.97D0*MDOTB2-MDOTGAS)*WTURB3+(.97D0*MDOTB1+.97D0*MDOTB2-
& MDOTGAS-MDOTF)*WTURB4
WPUMP=(MDOTB1+MDOTB2-MDOTF)*WPUMP1+MDOTB1*WPUMP2+MDOTB2*WPUMP3
WNET=WTURB-WPUMP
MDOT(39)=MDOTF
MDOT(24)=MDOT(18)
MDOT(25)=MDOT(18)
MDOT(26)=MDOT(18)
MDOT(27)=.97D0*MDOTB1+.97D0*MDOTB2-MDOTF-MDOTGAS
MDOT(28)=MDOTB1+MDOTB2-MDOTF
MDOT(29)=MDOT(28)+MDOT(39)
MDOT(30)=MDOTB1
MDOT(31)=MDOTB1
MDOT(32)=.97D0*MDOTB1
MDOT(33)=MDOT(32)
MDOT(34)=MDOT(33)
MDOT(35)=MDOT(34)-MDOT(21)
MDOT(36)=MDOT(35)
MDOT(37)=MDOT(36)+.97D0*MDOTB2
MDOT(38)=MDOT(37)
MDOT(40)=MDOT(38)-MDOT(39)
MDOT(41)=MDOT(40)
MDOT(42)=.03D0*MDOTB1+.03D0*MDOTB2+MDOTGAS
MDOT(43)=MDOT(27)+MDOT(42)
MDOT(44)=MDOTB2
MDOT(45)=MDOTB2
MDOT(46)=.97D0*MDOTB2
MDOT(47)=MDOT(46)
MDOT(48)=.03D0*MDOTB1
MDOT(49)=.03D0*MDOTB2
P(27)=P1
P(28)=P2
P(29)=P3
P(30)=P4
P(31)=P5
P(32)=P6
P(33)=P7
P(34)=P8
P(35)=P8
P(36)=P9
P(37)=P10
P(38)=P11
P(39)=P12
P(40)=P11
P(41)=P13
P(42)=P1
P(43)=P1
P(44)=P14
P(45)=P15
P(46)=P15
P(47)=P16
P(48)=P5
P(49)=P15
P(24)=P(18)
P(25)=P(18)
P(26)=P(18)
T(24)=TA2
T(25)=TA3
T(26)=TA4
T(27)=T1
T(28)=T2
T(29)=T3
T(30)=T4

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T(31)=T5
T(32)=T6
T(33)=T7
T(34)=T8
T(35)=T8
T(36)=T9
T(37)=T10
T(38)=T11
T(39)=T12
T(40)=T11
T(41)=T13
T(42)=T1
T(43)=T1
T(44)=T14
T(45)=T15
T(46)=T15
T(47)=T16
T(48)=T5
T(49)=T15
WTURB1=MDOT(33)*WTURB1
WTURB2=MDOT(35)*WTURB2
WTURB3=MDOT(37)*WTURB3
WTURB4=MDOT(40)*WTURB4
WPUMPR1=MDOT(28)*WPUMP1
WPUMPR2=MDOT(30)*WPUMP2
WPUMPR3=MDOT(44)*WPUMP3
QREJ=(.97D0*MDOTB1+.97D0*MDOTB2-MDOTF-MDOTGAS)*H13
& -(.97D0*MDOTB1+.97D0*MDOTB2-MDOTF-MDOTGAS)*HF1
QCONDENSER=QREJ
QBD=.03D0*MDOTB1*(HG5-HF1)+.03D0*MDOTB2*(HG15-HF1)
QGAS=MDOTGAS*(H8-HF1)-MDOTGAS*(H7-H8)
QPIPELOSS=.97D0*MDOTB1*(HG6-H7)+.97D0*MDOTB2*(HG15-H16)
QIN=MDOTAIR*CPAIR*(TA1-TA4)
QSUPER=MDOT(24)*(T(18)-T(24))*CPAIR
QHPBOIL=MDOT(25)*(T(24)-T(25))*CPAIR
QLPBOIL=MDOT(26)*(T(25)-T(26))*CPAIR
RETURN
98 WRITE(6,*)'THE TA3 IS WRONG FOR SOME REASON'
WRITE(6,119)TA3,TA3E
119 FORMAT(1X,'TEMP OF AIR AT BOILER #2 IS ',6X,F10.3,1X,'K',1X,'|',
&1X,F10.3,1X,'F')
RETURN
99 WRITE(6,*)'GUESS ANOTHER TA2'
RETURN
END
C
C
SUBROUTINE DTURBINE(NT,PENTER,PEXIT,TENTER,TEXIT,HENTER,HEXIT,
&SENER,SEXIT,WORKTURB)
DOUBLE PRECISION NT,PENTER,PEXIT,TENTER,TEXIT,HENTER,HEXIT,
&SENER,SEXIT,
&WORKTURB,XS,XR,VFX,VGX,HFX,HGX,SFX,SGX,TXS,VXS,HXS,VEXIT
CALL DSAT(TEXIT,PEXIT,VFX,VGX,HFX,HGX,SFX,SGX,2)
IF(SENER.GE.SGX)THEN
CALL DSUPER(TXS,PEXIT,VXS,HXS,SENER,2)
WORKTURB=NT*(HENTER-HXS)
HEXIT=HENTER-WORKTURB
CALL DSUPER(TEXIT,PEXIT,VEXIT,HEXIT,SEXIT,3)
ELSE
XS=(SENER-SFX)/(SGX-SFX)
HXS=XS*HGX+(1.0D0-XS)*HFX
WORKTURB=NT*(HENTER-HXS)
HEXIT=HENTER-WORKTURB
XR=(HEXIT-HFX)/(HGX-HFX)
SEXIT=XR*SGX+(1.0D0-XR)*SFX

```

```

ENDIF
RETURN
END

C
C
C   SATURATED AND SUPERHEATED STEAM TABLES
C
subroutine dlook(x1,x2,y1,y2,m,n,prop)
implicit double precision (a-h,o-z)
dimension prop(5),prop0(5)
count=0.
10 count=count+1
call dsuperprop(x1,x2,prop0)
b1=y1-prop0(m)
b2=y2-prop0(n)
dx=.01d0*x1
x1=x1+dx
call dsuperprop(x1,x2,prop)
a11=(prop(m)-prop0(m))/dx
a21=(prop(n)-prop0(n))/dx
x1=x1-dx
dx=.01d0*x2
x2=x2+dx
call dsuperprop(x1,x2,prop)
a12=(prop(m)-prop0(m))/dx
a22=(prop(n)-prop0(n))/dx
x2=x2-dx
dx2=(b2-a21*b1/a11)/(a22-a21*a12/a11)
dx1=(b1-a12*dx2)/a11
test=dabs(dx2/x2)+dabs(dx1/x1)
x2=x2+dx2
x1=x1+dx1
if(count.gt.10.d0) go to 20
if(test.gt..00001d0) go to 10
go to 30
20 write(6,600) x1,x2
600 format(" no converge, t=",e14.7," rho=",e14.7)
30 call dsuperprop(x1,x2,prop)
return
end
function drhoz(tr,pr)
implicit double precision (a-h,o-z)
if(tr-1.d0) 20,10,10
10 x=(pr/tr**3)**2
z=1.d0-x*(1.289d0/tr-.532d0*tr*x)
go to 30
20 z=1.d0-.7d0*(pr/tr)**2
30 drhoz=73.94d0*pr/z/tr
return
end
subroutine dstatprop(t,p,dpdt,rhol)
implicit double precision (a-h,o-z)
dimension d(8),f(8)
data (d(i),i=1,8)/3.6711257,-28.512396,222.6524,-882.43852,
*2000.276,-2612.2557,1829.7674,-533.5052/
data(f(i),i=1,8)/-7.41924,.29721,-.1155286,8.685635e-3,
* 1.094098e-3,-4.39993e-3,2.520658e-3,-5.218684e-4/
data a,tp,tc,pc,rhoc/.01,338.15,647.286,22089.,317./
x=a*(t-tp)
y=f(1)+x*(f(2)+x*(f(3)+x*(f(4)+x*(f(5)+x*(f(6)+x*(f(7)+
*x*(f(8)))))))
dydx=f(2)+x*(2.*f(3)+x*(3.*f(4)+x*(4.*f(5)+x*(5.*f(6)+
*x*(6.*f(7)+x*(7.*f(8))))))
p=pc*dexp((tc/t-1.)*y)

```

```

dpdt=p*(-y*tc/t/t+(tc/t-1.)*dydx*a)
x=(1.-t/tc)**(1./3.)
x3=x*x*x
rho=rhoc*(1.+x*(d(1)+x3*(d(4)+d(7)*x3)+x*(d(2)+x3*(d(5)+
*d(8)*x3)+x*(d(3)+d(6)*x3)))
return
end
subroutine dsat(t,p,vf,vg,hf,hg,sf,sg,n)
implicit double precision (a-h,o-z)
if(n-1) 10,10,20
10 call dsatprop(t,p,dpdt,rhol)
go to 30
20 t=4900./(17.572-dlog(p))
count=0.
25 count=count+1
call dsatprop(t,p0,dpdt,rhol)
t=t+(p-p0)/dpdt
err=dabs((p-p0)/p)
if(count.gt.10d0) go to 26
if(err.gt..00001d0) go to 25
go to 30
26 write(6,610) t
610 format(" no converge at tsat=",e14.7)
30 vf=1./rhol
call dsuper(t,p,vg,hg,sg,1)
hfg=t*(vg-vf)*dpdt
hf=hg-hfg
sf=sg-hfg/t
return
end
subroutine dsuper (t,p,v,h,s,n)
implicit double precision (a-h,o-z)
dimension prop(5),x(2),diff(2)
pr=p/22089.
if(n-2) 10,20,30
10 tr=t/647.29d0
x(1)=drhoz(tr,pr)
x(2)=.98*x(1)
count=0
15 count=count+1
do 16 i=1,2
call dsuperprop(t,x(i),prop)
16 diff(i)=prop(3)-p
x(2)=(x(2)*diff(1)-x(1)*diff(2))/(diff(1)-diff(2))
x(1)=.98*x(2)
if(count.gt.10) go to 17
test=dabs(diff(2)/p)
if(test.gt..00001d0) go to 15
h=prop(4)
s=prop(5)
go to 40
17 write(6,600) t,p
600 format(" did not converge at t=",e14.7," p=",e14.7)
go to 40
20 b=8.57-s-.4615*dlog(p/101.35)
c=-.2652-.4615*pr
xi=(-b+dsqrt(b*b-.8136*c))/-.4086
tr=xi**.33333
t=tr*647.29
rho=drhoz(tr,pr)
v=1./rho
call dlook(t,rho,p,s,3,5,prop)
t=prop(1)
h=prop(4)
go to 40

```



```

30  b=3072.3-h
    c=-418.2119*pr-130.2
    xi=(-b+sqrt(b*b-1210.6*c))/605.3
    tr=xi**.4
    t=tr*647.29
    rho=drhoz(tr,pr)
    v=1./rho
    call dlook(t,rho,p,h,3,4,prop)
    t=prop(1)
    s=prop(5)
40  v=1./prop(2)
    return
end
subroutine dsuperprop(t,rho,prop)
implicit double precision (a-h,o-z)
dimension prop(5),a(10,7),rhoa(7),c(7),g(6),cp(7),h(7),hi(7)
data(a(i,1),i=1,10)/2.9492937e-2,-1.3213917e-4,2.7464632e-7,
*-3.6093828e-10,3.4218431e-13,-2.4450042e-16,1.5518535e-19,
*5.9728487e-24,-.41030848,-4.1605860e-4/
data(a(i,2),i=1,10)/-5.198586e-3,7.7779182e-6,-3.3301902e-8,
*-1.6254622e-11,-1.7731074e-13,1.2748742e-16,1.3746153e-19,
*1.5597836e-22,.3373118,-2.0988866e-4/
data(a(i,3),i=1,10)/6.8335354e-3,-2.6149751e-5,6.532639e-8,
*-2.6181978e-11,0.,0.,0.,-1.3746618e-1,-7.3396848e-4/
data(a(i,4),i=1,10)/-1.5641040e-4,-7.2546108e-7,-9.2734289e-9,
*4.3125840e-12,0.,0.,0.,6.7874983e-3,1.0401717e-5/
data(a(i,5),i=1,10)/-6.3972405e-3,2.6409282e-5,-4.7740374e-8,
*5.6323130e-11,0.,0.,0.,-13687317,6.4581880e-4/
data(a(i,6),i=1,10)/-3.9661401e-3,1.5453061e-5,-2.9142470e-8,
*3.9568796e-11,0.,0.,0.,7.9847970e-2,3.9917570e-4/
data(a(i,7),i=1,10)/-6.9048554e-4,2.7407416e-6,-5.1028070e-9,
*3.9636085e-12,0.,0.,0.,1.3041253e-2,7.1531353e-5/
data(rhoa(j),j=1,7)/634.,1000.,1000.,1000.,1000.,1000.,1000./
data(g(i),i=1,6)/4.6e4,1.011249e3,.83893,-2.19989e-4,
*2.46619e-7,-9.7047e-11/
data e,ta,xm,tc,pc,rhoc,t0/.0048,1000.,18.016,647.286,22.089,
*317.,273.16/
r=8314.34/xm
s=6696.5776-g(1)/t+g(2)*dlog(t/t0)+t*(g(3)+t*(g(4)/2.+t*(g(5)/3.
*+t*(g(6)/4.)))+g(1)/t0-t0*(g(3)+t0*(g(4)/2.+t0*(g(5)/3.+t0*(g(6)
*/4.)))
u=2375020.7+g(1)*dlog(t/t0)+t*(g(2)+t*(g(3)/2.+t*(g(4)/3.+t*(
*g(5)/4.+t*(g(6)/5.)))-t0*(g(2)+t0*(g(3)/2.+t0*(g(4)/3.+t0*
*(g(5)/4.+t0*(g(6)/5.)))
tau=ta/t
tauc=ta/tc
c(1)=r*t
c(2)=c(1)*(tau-tauc)
cp(1)=r
temp=r*tau
cp(2)=c(2)/t-temp
do 10 i=3,7
c(i)=c(i-1)*(tau-2.5)
cp(i)=c(i)/t-temp*(tau-2.5+(i-2)*(tau-tauc))
10 temp=temp*(tau-2.5)
te=dexp(-e*rho)
do 40 j=1,7
temp=rho-rhoa(j)
h(j)=a(1,j)+a(2,j)*(temp+rho)
hi(j)=rho*(a(1,j)+a(2,j)*temp)
do 30 i=3,8
h(j)=h(j)+a(i,j)*temp*(rho-rhoa(j)+(i-1.)*rho)
temp=temp*(rho-rhoa(j))
30 hi(j)=hi(j)+a(i,j)*rho*temp
h(j)=h(j)+a(9,j)+rho*(2.*a(10,j)-e*a(9,j)-e*a(10,j)*rho))*te

```

```
h(j)=h(j)*rho*rho
40 hi(j)=hi(j)+rho*(a(9,j)+a(10,j)*rho)*te
   p=rho*r*t
   do 50 j=1,7
     p=p+c(j)*h(j)
     u=u+(c(j)-t*cp(j))*hi(j)
50   s=s-cp(j)*hi(j)
     s=s-r*dlog(rho)
     prop(1)=t
     prop(2)=rho
     prop(3)=p/1000.
     prop(4)=u/1000.+prop(3)/rho
     prop(5)=s/1000.
   return
end
```

## Graphics Program Listing

```

PROGRAM GRAPH3
INTEGER NAR
PARAMETER(NAR=20000)

C
C *****
C ** Program Variables **
C *****

REAL XLOCAL(20),YLOCAL(20),XMIN,XMAX,YMIN,YMAX,POINTS(NAR,3),
+ X(NAR),Y(NAR),XHORIZ(2),YHORIZ(2),XVERT(2),YVERT(2),
+ ORIG(NAR,4),TMAT(4,4),TEMP(NAR,4),TEST1(2,2),TEST2(2,2),
+ TMAT1(4,4),TMAT2(4,4),TMAT3(4,4),TMAT4(4,4),TMAT5(4,4),
+ TMAT6(4,4),TMAT7(4,4),TMAT8(4,4),ZLOCAL(20),PLOT(NAR,4),
+ PROJ(4,4),PLOT1(NAR,4),PLOT2(NAR,4),ZCMIN,ZCMAX,CLOW(255),
+ CHIGH(255),CFRACT,COLOR(255),PLOT3(NAR,4),PLOT4(NAR,4),
+ XSPOT,YSPOT,DELTAY,DELTAX,VAL2,VAL3,VAL4,VAL,PERCENT,DX,DY,
+ VAL1,VAL5,XCMIN,XCMAX,YCMAX,AVGXY,YCMIN,FACTOR,SCFACT,ANGLE,
+ BSHEAR,CSHEAR,DSHEAR,FSHEAR,HSHEAR,ISHEAR,RMAT,QMAT,MMAT,
+ LMAT,PMAT,MMAT
INTEGER ICOMP_NUM(NAR),IFIRST,ILAST,IDSEQ_NUM(NAR),CH,COL,LEG,
+ IDSEQ(NAR),OLD,HORIZ_PTS,VERT_PTS,IPOINT_NUM,FOUR,NUMCOLOR,
+ PRI,BX,N,I,M,P,FLAG2,L,J,FLAG1,NUM_DSEQ,NUM_COMPS,PLT1,PLT2
CHARACTER QUEST*32,QUEST2*32,HEADER_CORD*32,CHAR*9,TITLE*50,
+ HEADER_DSEQ*32,FILE1*32,FILE2*32,QUEST3*32,QUEST4*32,
+ TEXT*32,XAXIS*50,YAXIS*50,QUEST5*32,QUEST6*32
C *****
C ** LOAD IN COLOR DATA **
C *****
OPEN(UNIT=11,FILE='color.dat',STATUS='OLD')
READ(11,*) NUMCOLOR
DO 10 I=1,NUMCOLOR
READ(11,*) COLOR(I)
10 CONTINUE
CLOSE(11)
C----> Initial settings
FOUR=NAR
CH=0
COL=0

C
C *****
C ** Prompts for different functions **
C *****
100 DO 42 N=1,8
WRITE(6,*) ' '
42 CONTINUE
WRITE(6,*) ' Would you like to....'
WRITE(6,*) ' '
WRITE(6,*) ' A) Load object from files'
WRITE(6,*) ' B) Modify object'
WRITE(6,*) ' C) Adjust window'
WRITE(6,*) ' D) Color option'
WRITE(6,*) ' E) View/Print objects'
WRITE(6,*) ' '
WRITE(6,*) ' (Or "QUIT" to exit program)'
DO 43 N=1,8
WRITE(6,*) ' '
43 CONTINUE
READ(5,'(A32)') QUEST
C
C *****

```

```

C      ** Get points entered from a file **
C      *****
504  IF (QUEST.EQ.'A'.OR.QUEST.EQ.'a') THEN
      WRITE(6,*) ' '
      WRITE(6,*) ' Input the Coordinate File....'
      WRITE(6,*) ' Example: HOUSE.CORD'
      READ(5,'(A32)') FILE1
      OPEN(UNIT=10,FILE=FILE1,STATUS='OLD',ERR=504)
      READ(10,'(A32)') HEADER_CORD
      READ(10,*) NUM_COMPS
      DO 200 I=1,NUM_COMPS
200   READ(10,*) ICOMP_NUM(I),XLOCAL(I),YLOCAL(I),ZLOCAL(I)
      CONTINUE
      READ(10,*) IFIRST,ILAST
      DO 210 I=IFIRST,ILAST
      READ(10,*) IPOINT_NUM,POINTS(I,1),POINTS(I,2),
210   POINTS(I,3),ICOMP_NUM(I)
      CONTINUE
      CLOSE(10)
505  WRITE(6,*) ' '
      WRITE(6,*) ' Input the Sequence File'
      WRITE(6,*) ' Example: HOUSE.DSEQ'
      READ(5,'(A32)') FILE2
      OPEN(UNIT=11,FILE=FILE2,STATUS='OLD',ERR=505)
      READ(11,'(A32)') HEADER_DSEQ
      READ(11,*) NUM_DSEQ
      DO 220 I=1,NUM_DSEQ
220   READ(11,*) IDSEQ_NUM(I),IDSEQ(I)
      CONTINUE
      CLOSE(11)
C      **--> BECAUSE A DATA POINT IS REQUIRED TO BE IN THE FORM
C      DX,Y,Z,1] AND OUR DATA IS IN THE FORM [X,Y,Z,COMPONENT#],
C      THE LIST OF POINTS MUST BE CONVERTED.
      DO 81 I=IFIRST,ILAST
      ORIG(I,1)=POINTS(I,1)+XLOCAL(ICOMP_NUM(I))
      ORIG(I,2)=POINTS(I,2)+YLOCAL(ICOMP_NUM(I))
      ORIG(I,3)=POINTS(I,3)+ZLOCAL(ICOMP_NUM(I))
      ORIG(I,4)=1
      TEMP(I,1)=POINTS(I,1)+XLOCAL(ICOMP_NUM(I))
      TEMP(I,2)=POINTS(I,2)+YLOCAL(ICOMP_NUM(I))
      TEMP(I,3)=POINTS(I,3)+ZLOCAL(ICOMP_NUM(I))
      TEMP(I,4)=1
81   CONTINUE
      FLAG1=1
C      *****
C      ** FIND LIMITS **
C      *****
      XCMAX=-1000000000
      XCMIN=+1000000000
      DO 20 I=IFIRST,ILAST
      IF(ORIG(I,1).GT.XCMAX) XCMAX=ORIG(I,1)
      IF(ORIG(I,1).LT.XCMIN) XCMIN=ORIG(I,1)
20   CONTINUE
      YCMAX=-1000000000
      YCMIN=+1000000000
      DO 23 I=IFIRST,ILAST
      IF(ORIG(I,2).GT.YCMAX) YCMAX=ORIG(I,2)
      IF(ORIG(I,2).LT.YCMIN) YCMIN=ORIG(I,2)
23   CONTINUE
      ZCMAX=-1000000000
      ZCMIN=+1000000000
      DO 22 I=IFIRST,ILAST
      IF(ORIG(I,3).GT.ZCMAX) ZCMAX=ORIG(I,3)
      IF(ORIG(I,3).LT.ZCMIN) ZCMIN=ORIG(I,3)
22   CONTINUE

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```

WRITE(6,*) ' Do you wish to input your own color limits?'
WRITE(6,*) ' (Y or N)'
READ(5,'(A32)') QUEST6
IF (QUEST6.EQ.'Y'.OR.QUEST6.EQ.'y') THEN
  WRITE(6,*) 'Please input the MINimum Z value ...'
  READ(5,*) ZCHIN
  WRITE(6,*) 'Please input the MAXimum Z value ...'
  READ(5,*) ZCMAX
ENDIF
C *****
C ** SCALE Z AXIS **
C *****
AVGXY=(YCMAX+XCMAX-YCHIN-XCHIN)/2.
FACTOR=.2*(AVGXY/ZCMAX)
DO 24 I=IFIRST,ILAST
  ORIG(I,3)=ORIG(I,3)*FACTOR
  TEMP(I,3)=TEMP(I,3)*FACTOR
24 CONTINUE
C *****
C ** SET COLOR RANGE **
C *****
CFRACT=(ZCMAX-ZCHIN)/NUMCOLOR
DO 30 I=1,NUMCOLOR
  CLOW(I)=ZCHIN+((I-1)*CFRACT)
  CHIGH(I)=ZCHIN+(I*CFRACT)
30 CONTINUE
C *****
C ** SET INITIAL WINDOW SIZE **
C *****
XMIN=-.5
XMAX=2
YMIN=-1
YMAX=.5
ENDIF
C *****
C ** Object Modification **
C *****
IF (QUEST.EQ.'B'.OR.QUEST.EQ.'b') THEN
  IF (FLAG1.NE.1) THEN
    WRITE(6,*) ' '
    WRITE(6,*) ' You have attempted to make modifications'
    WRITE(6,*) ' to an object. However, because no object is'
    WRITE(6,*) ' in memory, you will be sent back to the'
    WRITE(6,*) ' main menu.'
    WRITE(6,*) ' '
    WRITE(6,*) ' (PRESS RETURN TO CONTINUE)'
    READ(5,*)
    FLAG2=1
  ENDIF
  IF (FLAG2.EQ.1) GOTO 21
  WRITE(6,*) ' MODIFY OBJECT'
  WRITE(6,*) ' '
  WRITE(6,*) ' You have chosen to make modifications to'
  WRITE(6,*) ' the object in memory. Please select the'
  WRITE(6,*) ' type of modification to be done.....'
  WRITE(6,*) ' '
  WRITE(6,*) ' A) Local Scaling'
  WRITE(6,*) ' B) Overall Scaling'
  WRITE(6,*) ' C) Shear'
  WRITE(6,*) ' D) Rotation'
  WRITE(6,*) ' E) Reflection'
  WRITE(6,*) ' F) Translation'
  WRITE(6,*) ' G) Projection'
  WRITE(6,*) ' '

```

```

WRITE(6,*) ' '
READ(5,'(A32)') QUEST2
C
C *****
C ** Local Scaling **
C *****
IF (QUEST2.EQ.'A'.OR.QUEST2.EQ.'a') THEN
CALL IDENT(TMAT1,4)
WRITE(6,*) ' '
WRITE(6,*) '          ---> LOCAL SCALING <---'
WRITE(6,*) ' '
WRITE(6,*) ' Within local scaling, objects can be
+ stretched (scaled) in either the X, Y or Z axis. Please enter
+ the axis to which you wish to scale...'
WRITE(6,*) ' '
WRITE(6,*) ' (NOTE: a value of zero will make no change)'
WRITE(6,*) ' '
WRITE(6,*) '          (Please type "X", "Y", or "Z")'
READ(5,'(A32)') QUEST3
IF (QUEST3.EQ.'X'.OR.QUEST3.EQ.'x') THEN
WRITE(6,*) ' '
WRITE(6,*) ' What is the scale factor?'
READ(6,*) SCFACT
TMAT1(1,1)=SCFACT
ENDIF
IF (QUEST3.EQ.'Y'.OR.QUEST3.EQ.'y') THEN
WRITE(6,*) ' '
WRITE(6,*) ' What is the scale factor?'
READ(6,*) SCFACT
TMAT1(2,2)=SCFACT
ENDIF
IF (QUEST3.EQ.'Z'.OR.QUEST3.EQ.'z') THEN
WRITE(6,*) ' '
WRITE(6,*) ' What is the scale factor?'
READ(6,*) SCFACT
TMAT1(3,3)=SCFACT
ENDIF
CALL TYPE(QUEST4)
IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN
CALL IDENT(TMAT,4)
CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)
CALL MULT(TEMP,FOUR,4,TMAT1,4,4)
ENDIF
IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN
CALL MULT(TEMP,FOUR,4,TMAT1,4,4)
ENDIF
CALL MULT(TMAT,4,4,TMAT1,4,4)
CALL NORMAL(TEMP,FOUR,4)
ENDIF
C
C *****
C ** Overall Scaling **
C *****
IF (QUEST2.EQ.'B'.OR.QUEST2.EQ.'b') THEN
CALL IDENT(TMAT2,4)
WRITE(6,*) ' '
WRITE(6,*) '          ---> OVERALL SCALING <---'
WRITE(6,*) ' '
WRITE(6,*) ' Within overall scaling, objects can be
+ stretched (scaled) in the X, Y, and Z axis at the same time.
+ Please enter the constant to which you wish to scale...'
WRITE(6,*) ' '
READ(6,*) SCFACT
SCFACT=1.0/SCFACT
TMAT2(4,4)=SCFACT

```

```

CALL TYPE(QUEST4)
IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN
  CALL IDENT(TMAT,4)
  CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)
  CALL MULT(TEMP,FOUR,4,TMAT2,4,4)
ENDIF
IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN
  CALL MULT(TEMP,FOUR,4,TMAT2,4,4)
ENDIF
CALL MULT(TMAT,4,4,TMAT2,4,4)
CALL NORMAL(TEMP,FOUR,4)
ENDIF
C
C *****
C ** Shearing **
C *****
IF (QUEST2.EQ.'C'.OR.QUEST2.EQ.'c') THEN
  CALL IDENT(TMAT3,4)
  WRITE(6,*) ' '
  WRITE(6,*) '          ----> SHEARING <----'
  WRITE(6,*) ' '
  WRITE(6,*) ' Within shearing, objects can be distorted
+ with respect to the X, Y, or Z axis. These constants are in the
+ form...'
  WRITE(6,*) ' '
  WRITE(6,*) '          | 1 B C 0 | '
  WRITE(6,*) '          | D 1 F 0 | '
  WRITE(6,*) '          | H I 1 0 | '
  WRITE(6,*) '          | 0 0 0 1 | '
  WRITE(6,*) ' '
  WRITE(6,*) '(NOTE: a value of zero will make no change.)'
  WRITE(6,*) ' '
  WRITE(6,*) ' What is the new "B" value'
  READ(5,*) BSHEAR
  WRITE(6,*) ' What is the new "C" value'
  READ(5,*) CSHEAR
  WRITE(6,*) ' What is the new "D" value'
  READ(5,*) DSHEAR
  WRITE(6,*) ' What is the new "F" value'
  READ(5,*) FSHEAR
  WRITE(6,*) ' What is the new "H" value'
  READ(5,*) HSHEAR
  WRITE(6,*) ' What is the new "I" value'
  READ(5,*) ISHEAR
  TMAT3(1,2)=BSHEAR
  TMAT3(1,3)=CSHEAR
  TMAT3(2,1)=DSHEAR
  TMAT3(2,3)=FSHEAR
  TMAT3(3,1)=HSHEAR
  TMAT3(3,2)=ISHEAR
  CALL TYPE(QUEST4)
IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN
  CALL IDENT(TMAT,4)
  CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)
  CALL MULT(TEMP,FOUR,4,TMAT3,4,4)
ENDIF
IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN
  CALL MULT(TEMP,FOUR,4,TMAT3,4,4)
ENDIF
CALL MULT(TMAT,4,4,TMAT3,4,4)
CALL NORMAL(TEMP,FOUR,4)
ENDIF
C
C *****
C ** Rotating **

```

```

C *****
  IF (QUEST2.EQ.'D'.OR.QUEST2.EQ.'d') THEN
    CALL IDENT(TMAT4,4)
    WRITE(6,*) ' '
    WRITE(6,*) '           ---> ROTATING <---'
    WRITE(6,*) ' '
    WRITE(6,*) ' Please enter the axis to which you wish
+ to rotate about (Type "X", "Y", or "Z")'
    WRITE(6,*) ' '
    READ(5,'(A32)') QUEST3
    WRITE(6,*) ' Please enter the angle (in degrees) to
+ which you want to rotate this object...'
    WRITE(6,*) ' '
    WRITE(6,*) 'NOTE: a positive angle will result in a
+ counter-clockwise rotation. Also, a value of zero will make
+ no change.'
    WRITE(6,*) ' '
    READ (5,*) ANGLE
    ANGLE=ANGLE*(3.14159/180)
    IF (QUEST3.EQ.'X'.OR.QUEST3.EQ.'x') THEN
      TMAT4(2,2)=COS(ANGLE)
      TMAT4(2,3)=SIN(ANGLE)
      TMAT4(3,2)=-SIN(ANGLE)
      TMAT4(3,3)=COS(ANGLE)
    ENDIF
    IF (QUEST3.EQ.'Y'.OR.QUEST3.EQ.'y') THEN
      TMAT4(1,1)=COS(ANGLE)
      TMAT4(1,3)=-SIN(ANGLE)
      TMAT4(3,1)=SIN(ANGLE)
      TMAT4(3,3)=COS(ANGLE)
    ENDIF
    IF (QUEST3.EQ.'Z'.OR.QUEST3.EQ.'z') THEN
      TMAT4(1,1)=COS(ANGLE)
      TMAT4(1,2)=SIN(ANGLE)
      TMAT4(2,1)=-SIN(ANGLE)
      TMAT4(2,2)=COS(ANGLE)
    ENDIF
    CALL TYPE(QUEST4)
    IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN
      CALL IDENT(TMAT,4)
      CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)
      CALL MULT(TEMP,FOUR,4,TMAT4,4,4)
    ENDIF
    IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN
      CALL MULT(TEMP,FOUR,4,TMAT4,4,4)
    ENDIF
    CALL MULT(TMAT,4,4,TMAT4,4,4)
    CALL NORMAL(TEMP,FOUR,4)
  ENDIF
C *****
C ** Reflection **
C *****
  IF (QUEST2.EQ.'E'.OR.QUEST2.EQ.'e') THEN
    CALL IDENT(TMAT5,4)
    WRITE(6,*) ' '
    WRITE(6,*) '           ---> REFLECTION <---'
    WRITE(6,*) ' '
    WRITE(6,*) ' Within reflection, objects can be
+ reflected in either the X axis, Y axis, or Z axis. Please enter
+ the axis to which you wish to reflect about...'
    WRITE(6,*) ' '
    WRITE(6,*) '           (Please type "X", "Y", or "Z")'
    READ(5,'(A32)') QUEST3
    IF (QUEST3.EQ.'X'.OR.QUEST3.EQ.'x') TMAT5(1,1)=-1

```



```

IF (QUEST3.EQ.'Y'.OR.QUEST3.EQ.'y') TMat5(2,2)=-1
IF (QUEST3.EQ.'Z'.OR.QUEST3.EQ.'z') TMat5(3,3)=-1
CALL TYPE(QUEST4)
IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN
  CALL IDENT(TMat,4)
  CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)
  CALL MULT(TEMP,FOUR,4,TMat5,4,4)
ENDIF
IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN
  CALL MULT(TEMP,FOUR,4,TMat5,4,4)
ENDIF
CALL MULT(TMat,4,4,TMat5,4,4)
CALL NORMAL(TEMP,FOUR,4)
ENDIF

C
C *****
C ** Translation **
C *****
IF (QUEST2.EQ.'F'.OR.QUEST2.EQ.'f') THEN
  CALL IDENT(TMat6,4)
  WRITE(6,*) ' '
  WRITE(6,*) ' ---> TRANSLATION <---'
  WRITE(6,*) ' '
  WRITE(6,*) ' Within translation, objects can be
+ moved up, down, right, or left. The relative constants are
+ in the form...'
  WRITE(6,*) ' '
  WRITE(6,*) ' | 1 0 0 0 | '
  WRITE(6,*) ' | 0 1 0 0 | '
  WRITE(6,*) ' | 0 0 1 0 | '
  WRITE(6,*) ' | L M N 1 | '
  WRITE(6,*) ' '
  WRITE(6,*) ' (NOTE: a value of zero will make no change.)'
  WRITE(6,*) ' '
  WRITE(6,*) ' What is the new "L" value'
  READ(5,*) LMat
  WRITE(6,*) ' What is the new "M" value'
  READ(5,*) MMat
  WRITE(6,*) ' What is the new "N" value'
  READ(5,*) NMat
  TMat6(4,1)=LMat
  TMat6(4,2)=MMat
  TMat6(4,3)=NMat
  CALL TYPE(QUEST4)
  IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN
    CALL IDENT(TMat,4)
    CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)
    CALL MULT(TEMP,FOUR,4,TMat6,4,4)
  ENDIF
  IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN
    CALL MULT(TEMP,FOUR,4,TMat6,4,4)
  ENDIF
  CALL MULT(TMat,4,4,TMat6,4,4)
  CALL NORMAL(TEMP,FOUR,4)
ENDIF

C
C *****
C ** Projection **
C *****
IF (QUEST2.EQ.'G'.OR.QUEST2.EQ.'g') THEN
  CALL IDENT(TMat7,4)
  WRITE(6,*) ' '
  WRITE(6,*) ' ---> PROJECTION <---'
  WRITE(6,*) ' '
  WRITE(6,*) ' Within projection, the object can take

```

+ on a variety of skewed looks. These constants are in  
 + the form...'

```
WRITE(6,*) ' |
WRITE(6,*) ' | 1 C 0 P |
WRITE(6,*) ' | 0 1 0 Q |
WRITE(6,*) ' | 0 0 1 R |
WRITE(6,*) ' | 0 0 0 1 |
```

WRITE(6,\*) ' (NOTE: a value of zero will make no change.)'

WRITE(6,\*) ' What is the new "P" value'

READ(5,\*) PMAT

WRITE(6,\*) ' What is the new "Q" value'

READ(5,\*) QMAT

WRITE(6,\*) ' What is the new "R" value'

READ(5,\*) RMAT

TMAT7(1,4)=PMAT

TMAT7(2,4)=QMAT

TMAT7(3,4)=RMAT

CALL TYPE(QUEST4)

IF (QUEST4.EQ.'A'.OR.QUEST4.EQ.'a') THEN

CALL IDENT(TMAT,4)

CALL CONVERT(ORIG,FOUR,4,TEMP,FOUR,4)

CALL MULT(TEMP,FOUR,4,TMAT7,4,4)

ENDIF

IF (QUEST4.EQ.'B'.OR.QUEST4.EQ.'b') THEN

CALL MULT(TEMP,FOUR,4,TMAT7,4,4)

ENDIF

CALL MULT(TMAT,6,6,TMAT7,4,4)

CALL NORMAL(TEMP,FOUR,4)

ENDIF

21 FLAG2=0

ENDIF

C

\*\*\*\*\*

C

\*\* Change Window \*\*

C

\*\*\*\*\*

390

IF (QUEST.EQ.'C'.OR.QUEST.EQ.'c') THEN

WRITE(6,\*) ' |

WRITE(6,\*) ' The present values of the world coordinates'

WRITE(6,\*) ' in the form (1,XMIN,XMAX,YMIN,YMAX) are...'

WRITE(6,\*) ' XMIN=',XMIN

WRITE(6,\*) ' XMAX=',XMAX

WRITE(6,\*) ' YMIN=',YMIN

WRITE(6,\*) ' YMAX=',YMAX

WRITE(6,\*) ' |

WRITE(6,\*) ' Would you like to change these ?'

WRITE(6,\*) ' (Type "Y" for yes or "N" to exit)'

READ(5,'(A32)') QUEST2

IF (QUEST2.EQ.'Y'.OR.QUEST2.EQ.'y') THEN

WRITE(6,\*) ' |

WRITE(6,\*) ' What is the new "XMIN" value'

READ(5,\*) XMIN

WRITE(6,\*) ' What is the new "XMAX" value'

READ(5,\*) XMAX

WRITE(6,\*) ' What is the new "YMIN" value'

READ(5,\*) YMIN

WRITE(6,\*) ' What is the new "YMAX" value'

READ(5,\*) YMAX

ENDIF

IF (QUEST2.EQ.'N'.OR.QUEST2.EQ.'n') GOTO 400

GOTO 390

400

ENDIF

C

\*\*\*\*\*

C

```

C   ** COLOR OPTION **
C   *****
      CH=0
      IF (QUEST.EQ.'D'.OR.QUEST.EQ.'d') THEN
        IF(COL.EQ.0) THEN
          WRITE(6,*) '   COLOR IS NOW ON'
          COL=1
          CH=1
        ENDIF
        IF(COL.EQ.1.AND.CH.EQ.0) THEN
          WRITE(6,*) '   COLOR IS NOW OFF'
          COL=0
          CH=1
        ENDIF
      CH=0
    ENDIF

C   *****
C   ** Drawing of objects and Text **
C   *****
      IF (QUEST.EQ.'E'.OR.QUEST.EQ.'e') THEN
        WRITE(6,*) ' '
        WRITE(6,*) ' '
        WRITE(6,*) ' '
        WRITE(6,*) '   Please enter the way in which you wish to
+ view this object...'
        WRITE(6,*) ' '
        WRITE(6,*) ' '
        WRITE(6,*) '           A)   Front View'
        WRITE(6,*) '           B)   Top View'
        WRITE(6,*) '           C)   Right Side View'
        WRITE(6,*) '           D)   Dimetric View'
        WRITE(6,*) '           E)   Isometric View'
        WRITE(6,*) ' '
        WRITE(6,*) ' '
        WRITE(6,*) ' '
        READ(5,'(A32)') QUEST3
        CALL IDENT(PROJ,4)
        CALL CONVERT(ORIG,FOUR,4,PLOT1,FOUR,4)
        CALL CONVERT(TEMP,FOUR,4,PLOT2,FOUR,4)
        IF(QUEST3.EQ.'A'.OR.QUEST3.EQ.'a') THEN
          PROJ(1,1)=0
          PLT1=2
          PLT2=3
        ENDIF
        IF(QUEST3.EQ.'B'.OR.QUEST3.EQ.'b') THEN
          PROJ(3,3)=0
          PLT1=2
          PLT2=1
          DO 501 I=1,ILAST
            PLOT1(I,1)=-PLOT1(I,1)
            PLOT2(I,1)=-PLOT2(I,1)
501      CONTINUE
        ENDIF
        IF(QUEST3.EQ.'C'.OR.QUEST3.EQ.'c') THEN
          PROJ(2,2)=0
          PLT1=1
          PLT2=3
          DO 502 I=1,ILAST
            PLOT1(I,1)=-PLOT1(I,1)
            PLOT2(I,1)=-PLOT2(I,1)
502      CONTINUE
        ENDIF
        IF(QUEST3.EQ.'D'.OR.QUEST3.EQ.'d') THEN
          PROJ(1,1)=.9428

```

```

PROJ(1,2)=.2425
PROJ(1,3)=.2287
PROJ(2,1)=.2357
PROJ(2,2)=.9701
PROJ(2,3)=.0572
PROJ(3,1)=.2357
PROJ(3,3)=.9718
PLT1=2
PLT2=3
ENDIF
IF(QUEST3.EQ.'E'.OR.QUEST3.EQ.'e') THEN
PROJ(1,1)=.577
PROJ(1,2)=.707
PROJ(1,3)=.408
PROJ(2,1)=.577
PROJ(2,2)=.707
PROJ(2,3)=.408
PROJ(3,1)=.577
PROJ(3,3)=.816
PLT1=2
PLT2=3
ENDIF
PRI=0
WRITE(6,*) ' Do you want to print this out?'
WRITE(6,*) ' (Y or N)'
READ(5,'(A32)') QUEST4
IF(QUEST4.EQ.'Y'.OR.QUEST4.EQ.'y') THEN
PRI=1
WRITE(6,*) 'Please input the TITLE'
READ(5,'(A40)') TITLE
WRITE(6,*) 'Please input the X-Axis lable'
READ(5,'(A40)') XAXIS
WRITE(6,*) 'Please input the Y-Axis lable'
READ(5,'(A40)') YAXIS
WRITE(6,*) 'Do you want the BOX drawn?'
WRITE(6,*) ' (Y OR N)'
READ(5,'(A32)') QUEST5
BX=0
IF(QUEST5.EQ.'N'.OR.QUEST5.EQ.'n') THEN
BX=0
ENDIF
IF(QUEST5.EQ.'Y'.OR.QUEST5.EQ.'y') THEN
BX=1
ENDIF
ENDIF
IF(QUEST4.EQ.'N'.OR.QUEST4.EQ.'n') THEN
PRI=0
ENDIF
WRITE(6,*) ' Do you want the Z-Axis drawn?'
WRITE(6,*) ' (Y or N)'
READ(5,'(A32)') QUEST4
IF(QUEST4.EQ.'N'.OR.QUEST4.EQ.'n') THEN
ZA=0
ENDIF
IF(QUEST4.EQ.'Y'.OR.QUEST4.EQ.'y') THEN
ZA=1
ENDIF
CALL MULT(PLOT1,FOUR,4,PROJ,4,4)
CALL MULT(PLOT2,FOUR,4,PROJ,4,4)
CALL NORMAL(PLOT1,FOUR,4)
CALL NORMAL(PLOT2,FOUR,4)
CALL CONVERT(PLOT2,FOUR,4,PLOT3,FOUR,4)
CALL CONVERT(PLOT2,FOUR,4,PLOT4,FOUR,4)
*****
C
C
** PRINT **

```

```

C *****
IF (PRI.EQ.1) THEN
  CALL PGBEGIN(0,'post.ps/ps',1,1)
  IF(BX.EQ.0) THEN
    CALL PGENV(XMIN,XMAX,YMIN,YMAX,1,-2)
  ENDIF
  IF(BX.EQ.1) THEN
    CALL PGENV(XMIN,XMAX,YMIN,YMAX,1,0)
  ENDIF
  CALL PGSCH(1.5)
  XSPOT=(XMIN+XMAX)/2.0
  YSPOT=YMAX+.07*(YMAX-YMIN)
  CALL PGPTXT (XSPOT,YSPOT,0.0,0.5,TITLE)
  CALL PGSCH(1.0)
  XSPOT=XMIN-.05*(XMAX-XMIN)
  YSPOT=(YMAX+YMIN)/2.0
  CALL PGPTXT (XSPOT,YSPOT,90.0,0.5,YAXIS)
  XSPOT=(XMIN+XMAX)/2.0
  YSPOT=YMIN-.11*(YMAX-YMIN)
  CALL PGSCH(1.0)
  CALL PGPTXT (XSPOT,YSPOT,0.0,0.5,XAXIS)
  CALL PGSCI(1)
  CALL PGSHLS(1,100.,1.,1.)
  OLD=0
  DO 509 I=1,MUM_DSEQ
    X(I-OLD)=PLOT4(ABS(IDSEQ(I)),PLT1)
    Y(I-OLD)=PLOT4(ABS(IDSEQ(I)),PLT2)
    IF(IDSEQ(I+1).GT.0..OR.IDSEQ(I).EQ.0.) GOTO 509
    CALL PGLINE((I-OLD),X,Y)
    OLD=I
509  CONTINUE
    CALL PGEND
  ENDIF

C *****
C ** OPEN PLOTTING PACKAGE **
C *****
CALL PGBEGIN(0,'/x',1,1)
IF(BX.EQ.0) THEN
  CALL PGENV(XMIN,XMAX,YMIN,YMAX,1,-2)
ENDIF
IF(BX.EQ.1) THEN
  CALL PGENV(XMIN,XMAX,YMIN,YMAX,1,0)
ENDIF
IF(PRI.EQ.1) THEN
  CALL PGSCH(1.5)
  XSPOT=(XMIN+XMAX)/2.0
  YSPOT=YMAX+.07*(YMAX-YMIN)
  CALL PGPTXT (XSPOT,YSPOT,0.0,0.5,TITLE)
  CALL PGSCH(1.0)
  XSPOT=XMIN-.05*(XMAX-XMIN)
  YSPOT=(YMAX+YMIN)/2.0
  CALL PGPTXT (XSPOT,YSPOT,90.0,0.5,YAXIS)
  XSPOT=(XMIN+XMAX)/2.0
  YSPOT=YMIN-.11*(YMAX-YMIN)
  CALL PGSCH(1.0)
  CALL PGPTXT (XSPOT,YSPOT,0.0,0.5,XAXIS)
ENDIF

C *****
C ** Graphics part of the program **
C *****
IF(COL.EQ.0) THEN
  CALL PGSCI(1)
  CALL PGSHLS(1,100.,1.,1.)

```

```

OLD=0
DO 508 I=1,NUM_DSEQ
  X(I-OLD)=PLOT3(ABS(IDSEQ(I)),PLT1)
  Y(I-OLD)=PLOT3(ABS(IDSEQ(I)),PLT2)
  IF(IDSEQ(I+1).GT.0..OR.IDSEQ(I).EQ.0.) GOTO 508
  CALL PGLINE((I-OLD),X,Y)
  OLD=I
508 CONTINUE
ENDIF
IF(COL.EQ.1) THEN
  WRITE(6,*) 'Do you wish to use the LEGEND?'
  WRITE(6,*) ' (Y OR N)'
  READ(5,'(A32)') QUEST5
  IF(QUEST5.EQ.'N'.OR.QUEST5.EQ.'n') THEN
    LEG=0
  ENDIF
  IF(QUEST5.EQ.'Y'.OR.QUEST5.EQ.'y') THEN
    LEG=1
  ENDIF
  IF(LEG.EQ.1) THEN
C *****
C ** LEGEND **
C *****
    CALL PGSLW(2)
    DELTAX=XMAX-XMIN
    DELTAY=YMAX-YMIN
    DX=DELTAX*.05
    PERCENT=.3
    DY=(DELTAY-PERCENT*DELTAY)/NUMCOLOR
    DO 60 I=1,NUMCOLOR
      Y(1)=(YMAX-.5*PERCENT*DELTAY)-(I-1)*DY
      Y(2)=(YMAX-.5*PERCENT*DELTAY)-(I-1)*DY
      Y(3)=(YMAX-.5*PERCENT*DELTAY)-(I)*DY
      Y(4)=(YMAX-.5*PERCENT*DELTAY)-(I)*DY
      X(1)=XMAX-.2*DELTAX
      X(2)=XMAX-.2*DELTAX+DX
      X(3)=XMAX-.2*DELTAX+DX
      X(4)=XMAX-.2*DELTAX
      L=NUMCOLOR+1-I
      VAL5=COLOR(L)
      CALL PGSHLS(L,VAL5,.5,1.0)
      CALL PGSCI(L)
      CALL PGSFS(1)
      CALL PGPOLY(4,X,Y)
      CALL PGLINE(4,X,Y)
      WRITE(CHAR,'(F8.3)') CHIGH(L)
      VAL1=X(2)+.5*DX
      VAL2=Y(2)-.2*DY
      CALL PGSCH(.8)
      CALL PGTEXT(VAL1,VAL2,CHAR)
60 CONTINUE
      WRITE(CHAR,'(F8.3)') CLOW(L)
      VAL1=X(2)+.5*DX
      VAL2=Y(3)-.3*DY
      CALL PGTEXT(VAL1,VAL2,CHAR)
    ENDIF
C *****
C ** DRAWING **
C *****
    L=2
    CALL PGSCI(2)
    CALL PGSLW(1)
    CALL PGSHLS(2,100.,1.,1.)
    OLD=0
    DO 507 I=1,NUM_DSEQ

```

```

X(I-OLD)=PLOT2(ABS(IDSEQ(I)),PLT1)
Y(I-OLD)=PLOT2(ABS(IDSEQ(I)),PLT2)
IF(IDSEQ(I+1).GT.0..OR.IDSEQ(I).EQ.0.) GOTO 507
IF(L.GT.250) L=2
VAL1=ORIG(ABS(IDSEQ(I-3)),3)
VAL2=ORIG(ABS(IDSEQ(I-2)),3)
VAL3=ORIG(ABS(IDSEQ(I-1)),3)
VAL4=ORIG(ABS(IDSEQ(I)),3)
VAL=(VAL1+VAL2+VAL3+VAL4)/(4.*FACTOR)
DO 50 J=1,NUMCOLOR
  IF (VAL.LE.CHIGH(J).AND.VAL.GT.CLOW(J))
    VAL5=COLOR(J)
50  CONTINUE
    CALL PGSHLS(L,VAL5,.5,1.0)
    CALL PGSCI(L)
    CALL PGSFS(1)
    CALL PGPOLY((I-OLD),X,Y)
    CALL PGLINE((I-OLD),X,Y)
    L=L+1
    OLD=I
507  CONTINUE
    ENDIF
C *****
C ** CLOSE PLOTTING PACKAGE **
C *****
    CALL PGEND
    ENDIF
IF (QUEST.EQ.'QUIT'.OR.QUEST.EQ.'quit') GOTO 999
GOTO 100
999  END
C *****
C ** Subroutines **
C *****
C
SUBROUTINE IDENT(MATRIX,SIZE)
INTEGER SIZE,I,J
REAL MATRIX(SIZE,SIZE)
DO 10 I=1,SIZE
  DO 20 J=1,SIZE
    MATRIX(I,J)=0
20  CONTINUE
10  CONTINUE
DO 30 I=1,SIZE
  MATRIX(I,I)=1
30  CONTINUE
RETURN
END
C
SUBROUTINE TYPE(TYPE_MOD)
CHARACTER TYPE_MOD*32
10  WRITE(6,*) ' '
WRITE(6,*) 'Do you wish to....'
WRITE(6,*) '  A) Modify original object'
WRITE(6,*) '  B) Modify current edited object'
WRITE(6,*) ' '
READ(5,'(A32)') TYPE_MOD
IF (TYPE_MOD.EQ.'A'.OR.TYPE_MOD.EQ.'a') RETURN
IF (TYPE_MOD.EQ.'B'.OR.TYPE_MOD.EQ.'b') RETURN
WRITE(6,*) 'Something other than "A" or "B" has been entered.'
WRITE(6,*) '          Please try again....'
GOTO 10
END
C
SUBROUTINE MULT(FIRST,ROW1,COL1,SECOND,ROW2,COL2)

```

```

        INTEGER ROW1,ROW2,COL1,COL2,I,J,K
        REAL FIRST(ROW1,COL1),SECOND(ROW2,COL2),THIRD(20000,4)
        DO 50 I=1,ROW1
            DO 60 J=1,COL2
                THIRD(I,J)=0
60         CONTINUE
50        CONTINUE
        IF (COL1.NE.ROW2) GOTO 10
        DO 20 K=1,COL1
            DO 30 I=1,ROW1
                DO 40 J=1,COL2
                    THIRD(I,J)=THIRD(I,J)+FIRST(I,K)*SECOND(K,J)
40         CONTINUE
30        CONTINUE
20        CONTINUE
        DO 70 I=1,ROW1
            DO 80 J=1,COL2
                FIRST(I,J)=THIRD(I,J)
80        CONTINUE
70        CONTINUE
        RETURN
10        WRITE (6,*) ' '
        WRITE (6,*) ' Because the length of column 1 does not equal the
+ length of row 2 these two matrices can not be multiplied
+ together!'
        WRITE(6,*) ' '
        WRITE (6,*) '          FATAL ERROR!'
        END
C
        SUBROUTINE PRARR2(POINTS,ROW,COL)
        INTEGER ROW,COL,I,J
        REAL POINTS(ROW,COL)
        DO 10 I=1,ROW
            DO 20 J=1,COL
                WRITE(6,*) '   ROW',I,'   COL',J
                WRITE(6,*) POINTS(I,J)
                WRITE(6,*) ' '
20         CONTINUE
10        CONTINUE
        RETURN
        END
C
        SUBROUTINE CONVERT(FIRST,ROW1,COL1,SECOND,ROW2,COL2)
        INTEGER ROW1,COL1,ROW2,COL2,I,J
        REAL FIRST(ROW1,COL1),SECOND(ROW2,COL2)
        DO 10 I=1,ROW1
            DO 20 J=1,COL1
                SECOND(I,J)=FIRST(I,J)
20        CONTINUE
10        CONTINUE
        RETURN
        END
C
        SUBROUTINE NORMAL(MAT,ROW,COL)
        INTEGER ROW,COL,I,J
        REAL MAT(ROW,COL)
        DO 20 I=1,ROW
            DO 30 J=1,COL
                IF (MAT(I,COL).NE.0) THEN
                    MAT(I,J)=MAT(I,J)/MAT(I,COL)
                ENDIF
30         CONTINUE
20        CONTINUE
10        RETURN
        END

```



C) Fixed Bed Coal Gasifier Model

MAIN PROGRAM

C

LOGICAL DEB, MON, OK, FPLOT, TPLOT

C

```
COMMON / D1 / RMOIST, VM, FIXC, ASH
COMMON / D2 / FC, FH, FO, FN, FS, FASH
COMMON / D3 / COAL, STEAM, AIR, OAIR, TCOAL, TSTEAM, TAIR
COMMON / D4 / IND, ITYPE
COMMON / D5 / HVCOAL, P, DIA, DP, VMD, PA
COMMON / D6 / EP, ETC, TW
COMMON / RR / RATE(6)
COMMON / XI / IC0, IC02, I02, IH2, ICH4, IH20, ICAR, IASH
COMMON / FF / F(8), AREA
COMMON / HE / H(9), CP(9)
COMMON / RHO / RHOC
COMMON / RD / C, CCOM, AC, YNUMBER
COMMON / DEB / OK, DEB, MON, FPLOT, TPLOT, METHOD
COMMON / PPP / PW2
COMMON / DE / THK, FR, DE, DENSO, CPP, DENW, CPW, HOVW
COMMON / DV / DGAS, HFTARV, HFTARL, HFGAS, CPTAR, CPGAS, DVTIME, HOV
IMPLICIT REAL*8 (A-H, O-Z)
CHARACTER ICHAR*7
DIMENSION I1(350), I3(350), I4(350), I5(350), I6(350), I7(350), I8(350)
DIMENSION Y1(350), ICHAR(7), I10(350)
DIMENSION PGAS(8), CS(350,10), HV(6), TAR(3), HR(6)
DIMENSION DF(8), X11(350), X12(350)
```

C

```
DATA (ICHAR(I), I=1,7) / '3333', '4444', '5555', '6666', '7777', '8888',
& 'TTTT' /
DATA DZ, TB, NSF, ISW, N, N154 / 1.0, 0.0, 3, 2, 8, 180 /
DATA PI, R / 3.14159, 82.057 /
DATA HV / 67636.0, 0.0, 0.0, 57798.0, 191759.0, 0.0 /
II=0
```

C

1 = T            2 = C            3 = CO            4 = CO2          5 = O2

C

6 = H2           7 = CH4           8 = H2O           9 = N2

C

```
NPLOT = 0
IRT = 0
111 IRT = IRT + 1
IF ( IRT .GT. 1 ) STOP
CALL INPUT
IF ( DEB ) WRITE (6,68)
```

C

C CONVERSION OF UNITS TO CGS SYSTEM

```

C
  COAL = COAL * 453.6 / 3600.0
  AIR = AIR * 453.6 / ( 3600.0 * ( ( 1.0 - OAIR ) * 28.0 +
+ OAIR * 32.0 ) )
  STEAM = STEAM * 453.6 / ( 3600.0 * 18.0 )
  TCOAL = ( TCOAL + 460.0 ) / 1.8
  TSTEAM = ( TSTEAM + 460.0 ) / 1.8
  TAIR = ( TAIR + 460.0 ) / 1.8
  TW = ( TW + 460.0 ) / 1.8
  P = ( P + 14.7 ) / 14.7
  HTC = HTC * 252.0 * 1.8 / ( 3600.0 * 30.48 * 30.48 )
  DP = DP * 2.54
  WRITE (6,1)COAL,AIR,STEAM,TCOAL,TSTEAM,TAIR,TW,P,HTC,DP
1  FORMAT (//,5X,'COAL,AIR,STEAM,TCOAL,TSTEAM,TAIR,TW,P,HTC,DP',
1/,5X,5G12.4,/,5X,5G12.4)
C
  IF ( DEB ) WRITE (6,10) COAL, STEAM, AIR, TCOAL, TSTEAM, TAIR,
+ P, HTC, TW
C
C TO EVALUATE OTHER CONSTANTS
C
  AREA = DIA * DIA * 30.48 * 30.48 * PI / 4.0
  CIR = PI * DIA * 30.48
  AC = ( 100.0 - VM * VMD - RMOIST - ASH ) / ( 100.0 * FC )
  FASH = ASH / 100.0
C
  IF ( DEB ) WRITE (6,12) AREA, CIR, AC
C
  DTTOP = 0.000
  DEVOLL = 0.000
C
  DO 2 ITERATIONS :
  FIRST ASSUME THE DEVOLATILIZATION LENGTH = 0.0 CM
  SECOND USE THE VALUE FOR DEVOL LENGTH OBTAINED IN THE FIRST CASE
C
  DO 550 IDEV=1,2
  C = COAL * FC
  AF = COAL * FASH
  CCOM = AIR * OAIR * 12.0
  CC = C * AC
  FN2 = AIR * ( 1.0 - OAIR )
  IF ( IND .EQ. 1 ) IC = PA
  IF ( IND .EQ. 2 ) HT = PA * 30.48
  IF ( IND .EQ. 2 ) IC = 0.99
C

```

C INLET GAS TEMPERATURE

C

CALL CPH ( 9, TAIR )  
CALL CPH ( 5, TAIR )  
CALL CPH ( 8, TSTEAM )

C

CPGIN = ( 1.0 - OAIR ) \* CP(9) + OAIR \* CP(5)  
CPMGIN = AIR \* CPGIN + STEAM \* CP(8)  
TGIN = ( AIR \* CPGIN \* TAIR + STEAM \* CP(8) \* TSTEAM ) / CPMGIN

C

IF ( DEB ) WRITE (6,16) TGIN

C

C INITIALIZATION

C

IT = 0

C

C BEGINNING OF OUTER LOOP OF ADJUSTING IC

C

100 IT = IT + 1

C

C DO THREE ITERATIONS ON THE GAS-SOLID TEMPERATURE DIFFERENCE

C

DO 495 IC=1,3  
TMAX = 0.0  
THL = 0.0  
Z = 0.0  
J = 1  
COUT = C \* ( 1.0 - IC )  
RTG = 0.0  
RTS = 0.0

C

F(1) = TGIN  
F(2) = COUT  
F(3) = 0.0  
F(4) = 0.0  
F(5) = AIR \* OAIR  
F(6) = 0.0  
F(7) = 0.0  
F(8) = STEAM  
FRAC = 1.000  
SUMHR=0.0

C

IF ( MON ) WRITE (6,20) IT , IC  
IF ( DEB ) WRITE (6,32) FI

C

C GRAPHICAL AND TABLE OUTPUT

C

```
SUMG = AIR + STEAM
Y1(J) = 0.0
X1(J) = TGIN
X3(J) = 0.0
X4(J) = 0.0
X5(J) = F(5) / SUMG
X6(J) = 0.0
X7(J) = 0.0
X8(J) = STEAM / SUMG
CF = F(2)
CPS = CPSOL (TGIN,CF,AF)
CALL CONV (TGIN,DP,HC)
X11(J)=HC*6.0DO*(1.0DO-EP)*AREA/DP/CPS/(CF+AF)
X12(J)=(1.0DO-FRAC)*SUMHR/CPS/(CF+AF)
```

C

```
DO 120 K = 2, 8
120 CS(J,K) = F(K) * 3600.0 / 453.6
CS(J,1) = F(1) * 1.8 - 460.0
CS(J,10) = X10(J) * 1.8DO - 460.0DO
140 CALL RUNGE( 8, F, DF, Z, DZ, KEY )
IF( KEY.NE.1 ) GO TO 340
```

C

C TOTAL GAS FLOW RATE

C

```
SUMG = FN2
```

C

```
DO 160 K = 3, 6
IF ( F(K) .LT. 1.0E-10 ) F(K) = 0.0
160 SUMG = SUMG + F(K)
```

C

```
IF ( F(8) .LT. 1.0E-10 ) F(8) = 0.0
SUMG = SUMG + F(7) + F(8)
```

C

C MOLE FRACTION OF GAS

C

```
ICO = F(3) / SUMG
ICO2 = F(4) / SUMG
IO2 = F(5) / SUMG
IH2 = F(6) / SUMG
ICH4 = F(7) / SUMG
IH2O = F(8) / SUMG
```

C

C TOTAL SOLID FLOW RATE

```

C
      SUMS = AF + F(2)
C
C MASS FRACTION OF SOLID
C
      ICAR = F(2) / SUMS
      XASH = AF / SUMS
C
C EVALUATE RATE CONSTANTS
C
      T = F(1)
      IJ = (J-1)*DZ
      TC = X10(J)+(Z-IJ)*(X10(J+1)-X10(J))/DZ
      IF(IC.EQ.1)TC=T
      CALL RATES1 ( TC , T)
C
      DO 180 K = 1, 2
180   IF ( RATE(K) .LT. 0.0 ) RATE(K) = 0.0
      DO 200 K = 4, 6
200   IF ( RATE(K) .LT. 0.0 ) RATE(K) = 0.0
C
C CO2
C
      T4 = 0.0
      IF ( RATE(3) .LE. 0.0 ) T4 = -RATE(3)
      TEST4 = ( RATE(5) + T4 ) * AREA * DZ
      IF ( TEST4 .LE. F(4) ) GO TO 220
      IF ( TEST4 .LE. 0.00001 ) GO TO 220
      RATE(5) = RATE(5) * F(4) / TEST4
      IF ( T4 .EQ. 0.0 ) GO TO 220
      RATE(3) = - T4 * F(4) / TEST4
C
C H2
C
220  TEST6 = ( T4 + RATE(4) + 2.0 * RATE(6) ) * AREA * DZ
      IF ( TEST6 .LE. F(6) ) GO TO 240
      IF ( TEST6 .LE. 0.00001 ) GO TO 240
      RATE(4) = RATE(4) * F(6) / TEST6
      RATE(6) = RATE(6) * F(6) / TEST6
      IF ( T4 .GT. 0.0 ) RATE(3) = RATE(3) * F(6) / TEST6
C
C CO
C
240  TEST3 = RATE(3) * AREA * DZ
      IF ( TEST3 .LE. F(3) ) GO TO 260

```

```
IF ( TEST3 .LE. 0.00001 ) GO TO 260
RATE(3) = F(3) * RATE(3) / TEST3
```

```
C
C H20
```

```
C
260 TEST8 = ( RATE(2) + RATE(3) ) * AREA * DZ
IF ( TEST8 .LE. F(8) ) GO TO 280
IF ( TEST8 .LE. 0.00001 ) GO TO 280
RATE(2) = F(8) * RATE(2) / TEST8
RATE(3) = F(8) * RATE(3) / TEST8
```

```
C
C 02
```

```
C
280 RATIO = 1.0
A5 = ( RATIO + 2.0 ) / ( 2.0 + 2.0 * RATIO )
TEST5 = ( A5 * RATE(1) + RATE(4) / 2.0 ) * AREA * DZ
IF ( TEST5 .LE. F(5) ) GO TO 300
IF ( TEST5 .LE. 0.00001 ) GO TO 300
RATE(1) = RATE(1) * F(5) / TEST5
RATE(4) = RATE(4) * F(5) / TEST5
```

```
C
C SPECIFIC HEAT OF COAL
```

```
C
300 CP(2) = 2.673 + 0.002617 * T - 116900.0 / ( T * T )
H(2) = 2.673*(T-298.) + 0.002617*(T*T-298.*298.)/2.
+ + 116900./T - 116900./298.
```

```
C
C CPMG = SUM OF ( MOLAR FLOW RATE * CP )
```

```
C
CALL CPH ( 9, T )
CPMG = CP(9) * FN2
```

```
C
DO 320 K = 3, 8
CALL CPH ( K, T )
CPMG = CPMG + CP(K) * F(K)
```

```
320 CONTINUE
```

```
C
C HEAT OF REACTION
```

```
C
RATIO = 1.0
HR(1) = ( H(4) + H(3) * RATIO - H(5) * ( 1.0 + RATIO/2. ) )
+ / ( 1.0 + RATIO ) - H(2)
HR(2) = H(3) + H(6) - H(8) - H(2)
HR(3) = H(4) + H(6) - H(3) - H(8)
HR(4) = H(8) - H(5) / 2.0 - H(6)
```

```
HR(5) = 2.0 * H(3) - H(4) - H(2)
HR(6) = H(7) - 2.0 * H(6) - H(2)
```

C

C DF

C

```
DF(2) = AREA * ( RATE(1) + RATE(2) + RATE(5) + RATE(6) ) * 12.0
DF(3) = AREA * ( RATE(1) * RATIO / ( 1.0 + RATIO ) + RATE(2)
+ - RATE(3) + 2.0 * RATE(5) )
DF(4) = AREA * ( RATE(1) / ( 1.0 + RATIO ) + RATE(3)
+ - RATE(5) )
DF(5) = AREA * ( - RATE(1) * ( 1.0 + RATIO / 2.0 ) / ( 1.0
+ + RATIO ) - RATE(4) / 2.0 )
DF(6) = AREA * ( RATE(2) + RATE(3) - RATE(4) - 2.0 * RATE(6) )
DF(7) = AREA * RATE(6)
DF(8) = AREA * ( RATE(4) - RATE(2) - RATE(3) )
```

C

C HEAT GENERATION

C

```
A8 = ( -HR(1) * RATE(1) - HR(3) * RATE(3) - HR(6) * RATE(6) )
+ * AREA
A9 = ( -HR(2) * RATE(2) - HR(5) * RATE(5) ) * AREA
A10 = - HR(4) * RATE(4) * AREA
```

C

C HEAT LOSS

C

```
HLOS = HTC * CIR * ( T - TW )
IF ( HLOS .LT. 0.0 ) HLOS = 0.0
```

C

C SPECIFIC HEAT OF SOLID

C

```
CF = F(2)
CPS = CPSOL (T,CF,AF)
```

C

C OVERALL HEAT TRANSFER COEFFICIENT

C

```
CALL CONV(T,DP,HC)
HASH=2.0DO*THK/DP*(YNUMBER/(1.0DO-YNUMBER))
HAC=1.0DO/(1.0DO/HC+1.0DO/HASH)
HTRANS = HAC*6.0DO*(1.0DO-EP)*AREA/DP*(T-TC)
```

C

C DF (1)

C

```
SUMHR = A8 + A9 + A10
TERM = DF(2)*CPS*(TC-T)
SUMHR = SUMHR + TERM
```

```

      DF(1) = ( -HTRANS - HLOS + FRAC * SUMHR) / CPMG
      GO TO 140
340   HLOS1= HTC * CIR * (F(1)-TW)
      THL = THL + (HTC * CIR * (F(1)-TW)) * DZ
C
      J = J + 1
      X11(J)=HAC*6.0DO*(1.0DO-EP)*AREA/DP/SUMS/CPS
      X12(J)=(1.0DO-FRAC)*SUMHR/SUMS/CPS
C
C  DEBUGGING
C
      IF ( .NOT. DEB ) GO TO 360
      WRITE (6,88) J, Z
      WRITE (6,86) RATE
      WRITE (6,84) HR
      WRITE(6,92) F
C
C  TOTAL GAS FLOW RATE
C
360   SUMG = FN2
      DO 380 K = 3, 8
      IF ( F(K) .LT. 0.0 ) F(K) = 0.0
380   SUMG = SUMG + F(K)
C
      T = F(1)
      IF ( T .GT. TMAX ) TMAX = T
      VEL = SUMG * 22400.0 * T / ( P * 273.0 * AREA )
      RTG = DZ / VEL + RTG
C
C  TOTAL SOLID FLOW RATE
C
      SUMS = AF + F(2)
      VEL = SUMS / ( RHOC * ( 1.0 - EP ) * AREA )
      RTS = DZ / VEL + RTS
C
C  FOR GRAPHICAL OUTPUT
C
400   Y1(J) = Z / 30.48
C
      X1(J) = T
      X3(J) = F(3) / SUMG
      X4(J) = F(4) / SUMG
      X5(J) = F(5) / SUMG
      X6(J) = F(6) / SUMG
      X7(J) = F(7) / SUMG

```



```

      X8(J) = F(8) / SUMG
C
C TABLE OUTPUT
C
      DO 420 K = 2, 8
420   CS(J,K) = F(K) * 3600.0 / 453.6
C
      CS(J,1) = F(1) * 1.8 - 460.0
      CS(J,10) = X10(J) * 1.8D0 - 460.0D0
C
      IF ( J .LT. 348 ) GO TO 460
C
C MAXIMUM LENGTH EXCEEDED
C
      Z = Z / 30.48
      WRITE (6,56) Z
      IF ( MON ) WRITE (6,60)
      WRITE (6,22) F
      WRITE (6,24) J, Z, SUMG, RTS, RTG, THL, TMAX
C
      GO TO 111
C
C TERMINATION CRITERION
C
460   IF ( IND .EQ. 2 ) GO TO 480
      IF ( F(2) .LT. CC ) GO TO 140
      GO TO 500
C
480   TESTZ=(HT-DEVOLL-Z)/DZ
      IF(TESTZ.GT..5)GO TO 140
      JS=J
      X10(J)=X1(J)-DTTOP
490   TERM1=(X11(J)+X11(J-1))*DZ/2.0D0
      TERM2=(X1(J-1)-X1(J))/TERM1-(X12(J)+X12(J-1))/2.0D0
      X10(J-1) = X1(J-1)-TERM2+(X10(J)-X1(J)+TERM2)/DEXP(TERM1)
C
      WRITE(6,*)'X10(J-1) = ',X10(J-1)
      J=J-1
      IF(J.GT.1)GO TO 490
      J=JS
495   CONTINUE
      WRITE(6,*) IT
C
C END OF INNER LOOP
C
C DEBUGGING

```

C  
500 IF ( .NOT. MON ) GO TO 520  
WRITE (6,22) F  
WRITE (6,24) J, Z, SUMG, RTS, RTG, THL, TMAX  
WRITE (6,23) CC ,F(2)

C  
C TERMINATION CRITERION FOR OUTER LOOP

C  
520 IF(IND.EQ.1) GO TO 540  
ER = CC - F(2)  
IF ( MON ) WRITE (6,38) IT, IC, ER.  
IF(ABS(ER).LT.1.0) GO TO 540  
CALL CRECT ( IT, IC, ER )  
IF ( .NOT. OK ) GO TO 111  
IF ( MON ) WRITE (6,26) IC  
IF ( IT .LT. 15 ) GO TO 100  
WRITE (6,40) IT  
GO TO 111

C  
C END OF OUTER LOOP

C  
C DEVOLATILIZATION

C  
540 WMI = 28.0 \* F(3) + 44.0 \* F(4) + 32.0 \* F(5) + 2.0 \* F(6) +  
+ 16.0 \* F(7) + 18.0 \* F(8) + 28.0 \* FN2  
WMI = WMI / SUMG  
IF ( MON ) WRITE (6,62) T, WMI

C  
CALL DEVO1 ( WMI, T, CMET, DCO, DCO2, DH2, DCH4, DH20, DN2,  
+ DH2S, DTAR, TAR, COUNT )  
IF ( COUNT .GT. 10.0 ) GO TO 111  
IF ( MON ) WRITE (6,18) DCO, DCO2, DH2, DCH4, DH20, DN2,  
+ DH2S, DTAR, TAR  
DMOIST = COAL \* RMOIST / ( 100.0 \* 18.0 )  
DCARBO = ( DCO + DCO2 + DCH4 ) \* 12.0  
AMW=78.0

C  
C GAS FLOW RATE AT TOP OF GASIFIER

C  
F(3) = F(3) + DCO  
F(4) = F(4) + DCO2  
F(6) = F(6) + DH2  
F(7) = F(7) + DCH4  
F(8) = F(8) + DMOIST + DH20  
F(2) = F(2) + DCARBO + TAR(1) + CMET \* 12.0

```

C
C
C GAS EXIT TEMPERATURE
C
  CF = C
  CPS = CPSOL ( TCOAL, CF, AF )
C
  CALL CPH ( 9, T )
  CPMG = CP(9) * ( FN2 + DN2 )
  DO 580 K = 3, 8
  CALL CPH ( K, T )
580  CPMG = CPMG + F(K) * CP(K)
  TO = TCOAL
  TF = T
  HFG = 578.0DO
  CALL CPH ( 8, TO )
  CPH20 = CP(8)/18.0DO
  DMOIST = RMOIST * COAL/100.0DO
  A1 = DTAR *(HFTARV-HFTARL-CPTAR*TO) + DGAS*(HFGAS-HFTARL-CPGAS*TO)
&    + DMOIST*(HFG - CPH20 * TO) + (DTAR + DGAS + DMOIST - COAL)
&    * CPS * TO
  B1 = DTAR*CPTAR + DGAS*CPGAS + DMOIST*CPH20
  C1 = (COAL - DTAR -DGAS - DMOIST ) * CPS
  DEVOLL = COAL * DVTIME / AREA
  HA = 6.0DO * DEVOLL * AREA * (1.0DO - EP) * HOV / DP
  CC = (CF+AF)*CPS/CPMG
  XNTU = HA/(CF +AF)/CPS
  GAMA = DEXP((CC-1.0DO)*XNTU)
  EFF = (1.0DO - GAMA)/(1.0DO-CC*GAMA)
  TCTOP = TO - EFF * (TO -TF)
  TE = (CPMG * TF -A1-C1*TCTOP)/(B1+CPMG)
C
C
  DTTOP = TF-TCTOP
  WRITE(6,*)'DVTIME = ',DVTIME,'SEC  COAL = ',COAL ,'GM PER SEC'
  WRITE(6,*)'TO      = ',TO      ,'K    TE   = ',TE    ,'K'
  WRITE(6,*)'TCTOP  = ',TCTOP  ,'K    TF   = ',TF   ,'K'
  WRITE(6,*)'DEVOLL = ',DEVOLL,'CM  DTTOP = ',DTTOP,'DELTA K'
550  CONTINUE
C
  OPEN(UNIT=22,FILE='NEWTEMP',STATUS='NEW')
  JTERP = J + 1
  TDEVOLL = DEVOLL/30.48DO
  Y1(JTERP)=Y1(J)+DEVOLL

```

```

DO 543, I=1,J
  X1(I)=X1(I)*1.8-460.0DO
543  X10(I)=X10(I)*1.8-460.0DO
  WRITE(22,*)JTERP,'      ! temperatures in F'
  DO 541 I = 1,J
541  WRITE(22,*)Y1(I),' ',X1(I)
  TEXTIT = TE *1.8DO - 460.0DO
  WRITE(22,*)Y1(JTERP),' ',TEXTIT
  WRITE(22,*)JTERP
  DO 542 I=1,J
542  WRITE(22,*)Y1(I),' ',X10(I)
  TENTER = TO *1.8DO - 460.0DO
  WRITE(22,*)Y1(JTERP),' ',TENTER
  CLOSE(22)

C
C
  SUMG = FN2 + DN2 + DH2S + DTAR / AMW
  DO 560 K = 3, 8
560  SUMG = SUMG + F(K)
  F(1) = TE
  T = TE

C
C GRAPHICAL OUTPUT
C
  J = J + 1
  Y1(J) = ( Z + DEVOLL ) / 30.48

C
  SUMG = SUMG - DTAR / AMW
  I1(J) = T * 1.8 - 460.0
  I3(J) = F(3) / SUMG
  I4(J) = F(4) / SUMG
  I5(J) = F(5) / SUMG
  I6(J) = F(6) / SUMG
  I7(J) = F(7) / SUMG
  I8(J) = F(8) / SUMG

C
C TABLE OUTPUT
C
  DO 660 K = 2, 8
660  CS(J,K) = F(K) * 3600.0 / 453.6
  CS(J,1) = T * 1.8 -460.0

C
C EVALUATION OF EFFICIENCY, ETC.
C
  STDE = 1.0 - ( F(8) - DMOIST - DH2O ) / STEAM

```

```

C
SUMG = SUMG - F(8) + DTAR / AMW
HVG = ( 94052.0 * TAR(1) + 57798.0 * TAR(2) ) / ( SUMG * AMW )
DO 680 K = 1, 6
680 HVG = HVG + HV(K) * F(K+2) / SUMG
HVG = HVG * 537.0 / ( 252.0 * 0.791 * 520.0 )
C
HVG1 = HVG * SUMG / ( SUMG + F(8) )
SUMG = SUMG - DTAR / AMW + F(8)
DO 700 K = 1, 6
PGAS(K) = F(K+2) * 100.0 / SUMG
700 CONTINUE
C
PGAS(7) = ( FN2 + DN2 ) * 100.0 / SUMG
PGAS(8) = DH2S * 100.0 / SUMG
C
C CPS = CPSOL ( TGIN )
C
SUMG = SUMG * 3600.0 / 453.6
SOUT = ( COUT + AF ) * 3600.0 / 453.6
COAL = COAL * 3600.0 / 453.6
THL = THL * 3600.0 / 252.0
T = T * 1.8 - 460.0
TGIN = TGIN * 1.8 - 460.0
TMAX = TMAX * 1.8 - 460.0
COLDE = HVG * SUMG * 347.46 * 100.0 / ( HVCOAL * COAL )
DO 720 K = 1, 4
720 TAR(K) = TAR(K) * 100.0 / DTAR
DTAR = DTAR * 3600.0 / 453.6
IF ( IND .EQ. 1 ) HT = Z / 30.48
SEXIT = X10(1) * 1.8 - 460.0
C
C OUTPUT
C
WRITE (6,66)
WRITE (6,44)
C
IF ( .NOT. TPLOT ) GO TO 145
OPEN(UNIT=11,FILE='NEWCO',STATUS='NEW')
OPEN(UNIT=12,FILE='NEWCO2',STATUS='NEW')
OPEN(UNIT=13,FILE='NEW02',STATUS='NEW')
OPEN(UNIT=14,FILE='NEWH2',STATUS='NEW')
OPEN(UNIT=15,FILE='NEWCH4',STATUS='NEW')
OPEN(UNIT=16,FILE='NEWH2O',STATUS='NEW')
WRITE(11,*)'      ',J

```

```
WRITE(12,*)'      ',J
WRITE(13,*)'      ',J
WRITE(14,*)'      ',J
WRITE(15,*)'      ',J
WRITE(16,*)'      ',J
```

C

```
DO 741 K=1,J,1
  WRITE(11,144)Y1(K),CS(K,3)
  WRITE(12,144)Y1(K),CS(K,4)
  WRITE(13,144)Y1(K),CS(K,5)
  WRITE(14,144)Y1(K),CS(K,6)
  WRITE(15,144)Y1(K),CS(K,7)
  WRITE(16,144)Y1(K),CS(K,8)
  FORMAT(1X,F6.3,4X,E20.6)
```

741

144

C

```
CLOSE(11)
CLOSE(12)
CLOSE(13)
CLOSE(14)
CLOSE(15)
CLOSE(16)
```

C

```
145 DO 740 K = 1, J, 5
    WRITE (6,46) Y1(K)
    WRITE (6,48) ( CS(K,M), M = 1, 8 )
```

740

C

```
CONTINUE
IF ( K .EQ. J ) GO TO 760
WRITE (6,46) Y1(J)
WRITE (6,48) ( CS(J,M), M = 1, 8 )
```

C

```
760 WRITE (6,50) PGAS
    WRITE (6,54) TAR
    WRITE (6,52) SUMG, HVG, HVG1, T, DTAR, SOUT, SEXIT, THL,
+ COLDE, STDE, TMAX
IF ( IND .EQ. 1 ) WRITE (6,58) HT
IF ( IND .EQ. 2 ) WRITE (6,64) IC
```

C

```
IF ( NPLOT .NE. 1 ) GO TO 780
```

C

C

C

C

C

C

```
WRITE (6,70)
CALL FPLOT ( X1, Y1, J, ICHAR(7), 1, J )
WRITE (6,72)
CALL FPLOT ( X3, Y1, J, ICHAR(1), 1, J )
WRITE (6,74)
```

```

C      CALL FP PLOT ( X4, Y1, J, ICHAR(2), 1, J )
C      WRITE (6,76)
C      CALL FP PLOT ( X5, Y1, J, ICHAR(3), 1, J )
C      WRITE (6,78)
C      CALL FP PLOT ( X6, Y1, J, ICHAR(4), 1, J )
C      WRITE (6,82)
C      CALL FP PLOT ( X7, Y1, J, ICHAR(5), 1, J )
C      WRITE (6,80)
C      CALL FP PLOT ( X8, Y1, J, ICHAR(6), 1, J )
C
780   GO TO 111
10    FORMAT ( ' COAL, STEAM, AIR, TCOAL, TSTEAM, TAIR ' /
+ 1X, 6E20.6 / ' P, HTC, TW = ', 3E20.6 )
12    FORMAT ( ' AREA, CIR, AC ', 3E20.6 )
16    FORMAT ( ' TGIN = ', E20.6 )
18    FORMAT ( ' DEVOLATILIZATION RESULT - CO, CO2, H2, CH4, H2O, ',
+ ' N2, H2S ' / 7E15.6 / 27X, ' TAR, C, H, O, N ' / 5E15.6 )
20    FORMAT ( / ' OUTER LOOP ITERATION ', I5, 4X, ' TEMPERATURE ITERATION ', I1 )
22    FORMAT ( ' F ', 8E15.6 )
23    FORMAT ( ' CC= ', E15.6, 10X, ' F(2)= ', E15.6 )
24    FORMAT ( ' J, Z, SUMG, RTS, RTG, THL, TMAX' / I5, 6E15.6 )
26    FORMAT ( ' UPDATED XC = ', E15.6 )
32    FORMAT ( ' FI ', 8E15.6 )
38    FORMAT ( ' OUTER LOOP ITERATION - IT, XC, ER ', I5, 2E15.6 )
40    FORMAT ( // ' OUTER LOOP LIMIT OF ', I5, ' ITERATIONS EXCEEDED ' )
44    FORMAT ( / ' PROFILES OF FLOW RATES AND TEMPERATURES ',
+ ' (UNITS ARE F (1), LB/HR (2), LBMOLE/HR (3-8)) ' /
+ 5X, ' Z', 11X, ' T', 19X, ' C', 18X, ' CO', 17X, ' CO2' /
+ 16X, ' O2', 18X, ' H2', 17X, ' CH4', 17X, ' H2O' )
46    FORMAT ( 1X, F6.3 )
48    FORMAT ( 6X, 4E20.6 / 6X, 4E20.6 )
50    FORMAT ( / ' PERCENTAGE OF GAS PRODUCT ' / 4X, ' CO',
+ 7X, ' CO2', 8X, ' O2', 8X, ' H2', 7X, ' CH4', 7X, ' H2O', 8X, ' N2',
+ 7X, ' H2S' / 8F10.5 )
52    FORMAT ( / ' TOTAL GAS OUTPUT          = ', E15.6, ' LBMOLE/HR' /
+ ' HEATING VALUE OF GAS (DRY) = ', E15.6, ' BTU/SCF' /
+ ' HEATING VALUE OF GAS (WET) = ', E15.6, ' BTU/SCF' /
+ ' GAS EXIT TEMPERATURE          = ', E15.6, ' F' /
+ ' AMOUNT OF TAR                  = ', E15.6, ' LB/HR' //
+ ' TOTAL SOLID OUTPUT             = ', E15.6, ' LB/HR' /
+ ' SOLID EXIT TEMP                = ', E15.6, ' F' //
+ ' TOTAL HEAT LOSS                = ', E15.6, ' BTU/HR' /
+ ' THERMAL EFFICIENCY             = ', E15.6, ' PERCENT' //
+ ' STEAM DECOMPOSITION           = ', E15.6, /
+ ' MAXIMUM TEMPERATURE           = ', E15.6, ' F' // )

```

```

54  FORMAT ( / ' PERCENTAGE OF TAR PRODUCT ' / 4X, 'C', 9X, 'H',
+ 9X, 'O', 9X, 'N' / 4F10.4 )
56  FORMAT ( // ' MAXIMUM LENGTH OF ', F6.1, ' FT EXCEEDED. ',
+ ' INCREASE BED DIAMETER OR DECREASE CARBON CONVERSION ',
+ ' AND TRY AGAIN. ' )
58  FORMAT ( ' BED HEIGHT           = ', E15.6, ' HT ' )
60  FORMAT ( ' THE FOLLOWINGS ARE THE VALUES AT THE LAST COMPARTMENT' )
62  FORMAT ( ' DEVOLATILIZATION - T, WMI = ', 2E15.6 )
64  FORMAT ( ' CARBON CONVERSION     = ', E15.6 )
66  FORMAT ( / 1X, 11('*') // 1X, 'OUTPUT DATA' // 1X, 11('*') // )
68  FORMAT ( // 1X, 9('*') // 1X, 'DEBUGGING' // 1X, 9('*') // )
70  FORMAT ( 1H1, 20X, 'TEMPERATURE (F) VS. DISTANCE (FT) ' )
72  FORMAT ( 1H1, 20X, 'MOLE FRACTION OF CO VS. DISTANCE (FT) ' )
74  FORMAT ( 1H1, 20X, 'MOLE FRACTION OF CO2 VS. DISTANCE (FT) ' )
76  FORMAT ( 1H1, 20X, 'MOLE FRACTION OF O2 VS. DISTANCE (FT) ' )
78  FORMAT ( 1H1, 20X, 'MOLE FRACTION OF H2 VS. DISTANCE (FT) ' )
80  FORMAT ( 1H1, 20X, 'MOLE FRACTION OF H2O VS. DISTANCE (FT) ' )
82  FORMAT ( 1H1, 20X, 'MOLE FRACTION OF CH4 VS. DISTANCE (FT) ' )
84  FORMAT ( ' HR      ', 6E15.6 )
86  FORMAT ( ' RATE   ', 6E15.6 )
88  FORMAT ( ' J ', 15, ' Z ', E15.6 )
92  FORMAT ( ' F ', 8E15.6 )

```

C

END



## ORIGINAL DEVO SUBROUTINE

SUBROUTINE DEVO ( WMI, T, CMET, DCO, DCO2, DH2, DCH4, DH20, DN2,  
+ DH2S, DTAR, TAR )

C

LOGICAL OK, DEB, MON, TPLLOT, FPLLOT  
DIMENSION TAR(3)  
COMMON / D1 / RMOIST, VM, FIXC, ASH  
COMMON / D2 / FC, FH, FO, FN, FS, FASH  
COMMON / D3 / COAL, STEAM, AIR, OAIR, TCOAL, TSTEAM, TAIR  
COMMON / FF / F(8)  
COMMON / RD / C, CCOM, AC  
COMMON / DEB / OK, DEB, MON, FPLLOT, TPLLOT, METHOD  
IMPLICIT REAL\*8 (A-H,O-Z)

C

TO = TCOAL  
TT = T  
IF ( FPLLOT .OR. TPLLOT ) THEN  
    WRITE(6,\*) ' CALLING DEVOLP '  
    CALL DEVOLP ( WMI, T, TO, DTAR, DGAS )  
ELSE  
    WRITE(6,\*) ' CALLING DEVOL '  
    CALL DEVOL ( WMI, T, TO, DTAR, DGAS )  
ENDIF

C

DTAR=0.2

C

DGAS=0.5

T = TT  
AMF = ( VM + FIXC ) / 100.0  
DGAS = DGAS \* AMF \* COAL  
DTAR = DTAR \* AMF \* COAL  
TOTAL = DGAS + DTAR

C

CDEVOL = TOTAL - ( FH + FO + FN + FS ) \* COAL  
CMET = ( 1.0 - AC ) \* FC \* COAL - CDEVOL  
IF ( CMET .GE. 0.0 ) GO TO 100  
DGAS = ( TOTAL + CMET ) \* DGAS / TOTAL  
DTAR = ( TOTAL + CMET ) \* DTAR / TOTAL  
TOTAL = TOTAL + CMET  
CMET = 0.0  
GO TO 200

C

100 CMET = CMET / 12.0  
F(7) = F(7) + CMET  
F(6) = F(6) - CMET \* 2.0

C

200

GAM = FH \* COAL

C

TOTAL = DTAR / ( FC + FH + FO )

TAR(1) = FC \* TOTAL

TAR(2) = FH \* TOTAL

TAR(3) = FO \* TOTAL

C

DN2 = GAM / 28.0

DH2S = FS \* COAL / 32.0

C

GH = FH \* COAL - TAR(2) - 2.0 \* DH2S

GOXY = FO \* COAL - TAR(3)

GC = DGAS - GOXY - GH - DN2 \* 28.0 - DH2S \* 34.0

C

R = GC / ( F(3) + F(4) + F(7) )

R = R / 12.0

DCO = F(3) \* R

DCO2 = F(4) \* R

DCH4 = F(7) \* R

DH2O = GOXY / 16.0 - DCO - DCO2 \* 2.0

DH2 = ( GH - 2.0 \* DH2O - 4.0 \* DCH4 ) / 2.0

C

RETURN

END

D) Coal Gasifier Devolatilization Subroutines

**MODIFIED DEVO SUBROUTINE**

SUBROUTINE DEVO1 ( WMI, T, CMET, DCO, DCO2, DH2, DCH4, DE20, DN2,  
+ DH2S, DTAR, TAR, COUNT )

C

DIMENSION TAR(3), I(2), DIFF(2)  
IMPLICIT REAL\*8 (A-H,O-Z)  
LOGICAL OK, DEB, MON, FPLOT, TPLOT  
COMMON / D1 / RMOIST, VM, FIXC, ASH  
COMMON / D2 / FC, FH, FO, FN, FS, FASH  
COMMON / D3 / COAL, STEAM, AIR, OAIR, TCOAL, TSTEAM, TAIR  
COMMON / FF / F(8), AREA  
COMMON / RD / C, CCOM, AC, YNUMBER  
COMMON / ROLL / NDX  
COMMON / DEB / OK, DEB, MON, FPLOT, TPLOT, METHOD  
COMMON / DV / DDGAS, HFTARV, HFTARL, HFGAS, CPTAR, CPGAS, DVTIME, HOV

C

NDX= 10  
TO = TCOAL  
TT = T  
ADEVOL = (1.0DO - AC)\*FC\*COAL  
I(1) = 1.3DO\*VM/(VM+FC)+.025DO  
I(2) = .95DO\*I(1)  
COUNT = 0.0DO  
COUNT = COUNT + 1.0DO

5

C

CALL DEVOL1 ( WMI, T, TO, DTAR, DGAS, I(I) )

C

T = TT  
AMF = ( VM + FIXC ) / 100.0  
DGAS = DGAS \* AMF \* COAL  
DDGAS = DDGAS \* AMF \* COAL  
DTAR = DTAR \* AMF \* COAL  
TOTAL = DGAS + DTAR  
CDEVOL = TOTAL - ( FH + FO + FN + FS ) \* COAL

C

10

DIFF(I) = CDEVOL - ADEVOL  
I(2)=(I(2)\*DIFF(1)-I(1)\*DIFF(2))/(DIFF(1)-DIFF(2))  
I(1)=.98DO\*I(2)  
IF(COUNT.GT.10)GO TO 20  
TEST=DABS(DIFF(2))/ADEVOL  
IF(TEST.GT.1D-4)GO TO 5

C

200

C

GAN = FN \* COAL

```

TOTAL = DTAR / ( FC + FH + FO )
TAR(1) = FC * TOTAL
TAR(2) = FH * TOTAL
TAR(3) = FO * TOTAL
C
DN2 = GAN / 28.0
DH2S = FS * COAL / 32.0
C
GH = FH * COAL - TAR(2) - 2.0 * DH2S
GOXY = FO * COAL - TAR(3)
GC = DGAS - GOXY - GH - DN2 * 28.0 - DH2S * 34.0
C
R = GC / ( F(3) + F(4) + F(7) )
R = R / 12.0
DCO = F(3) * R
DCO2 = F(4) * R
DCH4 = F(7) * R
DH20 = GOXY / 16.0 - DCO - DCO2 * 2.0
DH2 = ( GH - 2.0 * DH20 - 4.0 * DCH4 ) / 2.0
C
GO TO 30
20 WRITE(6,300)
30 RETURN
300 FORMAT(3X,'UNABLE TO FIND SUITABLE X1 IN SUBROUTINE DEVO')
END

```

## MODIFIED DEVOL SUBROUTINE

```
SUBROUTINE DEVOL(WMI,TEMP,TEMO,TTWLV,TTWLG,X1)
LOGICAL DEB, OK, MON, FPLOT, TPLOT
IMPLICIT REAL*8 (A-H,O-Z)
REAL*8 YY(1),TWLVVV1(20),TWLVVV2(20),TIME(20),B2(11),B3(11)
REAL*8 K1(11),K2(11),K3(11),K4(11),A(11),B1(11),KG
REAL*8 C(11),D1(11),D2(11),D3(11),R(11),R1(11),R2(11),R3(11)
REAL*8 Y1(11),Y2(11),Y3(11),Y4(11),Y5(11),Y6(11)
REAL*8 QRTP(11),QYP(7,11),TWLVVV3(20),TWLVVV4(20),CCOAL(11)
REAL*8 K5(11),Y7(11),A4(11),B4(11),C4(11),D4(11),CK1(11)
REAL*8 CK2(11),CK3(11),K6(11),K7(11),K8(11),K9(11),K10(11)
COMMON / D1 / RMOIST, VM1, FIXC, ASH
COMMON / D4 / JND, ITYPE
COMMON / D5 / HVCOAL, P, DIA, DP1, VMD, PA
COMMON / DE / THK, FR, DR, DENSO, CP, DENW, CPW, HOVW
COMMON / CC / DP
COMMON / DEB / OK, DEB, MON, FPLOT, TPLOT, METHOD
COMMON / ROLL / NDI
COMMON / DV / DDGAS, HFTARV, HFTARL, HFGAS, CPTAR, CPGAS, DVTIME, HOV
```

C

C

CONSTANTS FOR TEMPERATURE PROFILE

C

```
HFGAS= -940.46
HFTARV = -1429.1
HFTARL = -1497.9
HFINERT = -85.41
CPTAR = 0.98D0
CPGAS = 0.98D0
CPINERT = 0.27D0
COALK=6.2D-4
DP = DP1
VM = VM1 / ( VM1 + FIXC )
```

C

C

INITIALZE CONSTANTS

C

```
NDIP=NDI+1
DO 1 I=1,NDIP
  K1(I)=0.0D0
  K2(I)=0.0D0
  K3(I)=0.0D0
  K4(I)=0.0D0
  K5(I)=0.0D0
  K6(I)=0.0D0
  K7(I)=0.0D0
```

```

      K8(I)=0.0DO
      K9(I)=0.0DO
1     K10(I)=0.0DO
C
      TEMAV = ( TEMO + TEMP)/2.0DO
      CALL CONV ( TEMAV, DP, HOV )
      STEP = DP/100.0DO/HOV
C
      SIGM=1.355D-12
      DEFF=5.0D-3
      CNSTW=HOVW/CPW/DENW
      COAL0=1.0DO
      WGB=0.0DO
      WVB=0.0DO
      WVS=0.0DO
      WGS=0.0DO
      SHN=2.0DO+6.024D0*DP**0.5D0
      DEML=7.2D-2
      KG=DEML*SHN/DP
      DX=DP/2.0DO/FLOAT(NDX)
      WIB=P*WMI/82.06D0/TEMP
      R(1)=DX/2.0DO
      R(NDXP)=DP/2.0DO
      K1(1)=3.0DO*STEP/R(1)**3.0DO
      K2(1)=R(1)**2.0DO
      K5(1)=(R(1)*R(1)*R(1))*STEP/3.0DO
DO 45 I=2,NDX
45     R(I)=R(I-1)+DX
DO 451 I=1,NDXP
451    QRTP(I)=R(I)
DO 40 I=2,NDXP
      K1(I)=3.0DO*STEP/(R(I)**3.0DO-R(I-1)**3.0DO)
      K2(I)=R(I)**2.0DO
40     K5(I)=(R(I)**3.0DO-R(I-1)**3.0DO)*STEP/3.0DO
DO 41 I=1,NDXP
      CCOAL(I)=COAL0
      Y1(I)=0.0DO
      Y2(I)=0.0DO
      Y3(I)=0.0DO
      Y4(I)=0.0DO
      Y5(I)=0.0DO
      Y6(I)=WIB
41     Y7(I)=TEMO
      TWLV=0.0
      TWLG=0.0

```

```

TEMWP=0.0
TEMW=TEM0
CALL CONSTK1(ITYPE,CK10,CK20,CK30,E1,E2,E3)
CK20 = 2.9D12
E2 = 49200.0D0
CK30 = CK20
E3 = E2
DO 3 J=1,1000
  T=STEP*FLOAT(J)
  DO 990 I=1,NDXP
    CK1(I)=CK10*DEXP(-E1/1.987/Y7(I))
    CCOAL(I)=CCOAL(I)*DEXP(-CK1(I)*STEP)
    CK2(I)=CK20*DEXP(-E2/1.987/Y7(I))
990    CK3(I)=CK30*DEXP(-E3/1.987/Y7(I))
    K3(1)=(Y1(1)+Y2(1)+Y3(1))/(Y4(1)+Y5(1)+Y6(1)+
    &      Y4(2)+Y5(2)+Y6(2))
    K4(1)=0.0D0
    DO 421 I=2,NDX
      K3(I)=(Y1(I)+Y2(I)+Y3(I))/(Y4(I)+Y5(I)+Y6(I)+
    &      Y4(I+1)+Y5(I+1)+Y6(I+1))
      K4(I)=(Y1(I-1)+Y2(I-1)+Y3(I-1))/(Y4(I)+Y5(I)+Y6(I)+
    &      Y4(I-1)+Y5(I-1)+Y6(I-1))
421    CONTINUE
    K3(NDXP)=(Y1(NDXP)+Y2(NDXP)+Y3(NDXP))/(Y4(NDXP)+Y5(NDXP)+
    &      Y6(NDXP))
    K4(NDXP)=(Y1(NDX)+Y2(NDX)+Y3(NDX))/(Y4(NDXP)+Y5(NDXP)+Y6(NDXP)+
    &      Y4(NDX)+Y5(NDX)+Y6(NDX))
C
    A(1)=0.0D0
    B1(1)=1.0D0+(CK2(1)+CK3(1))*STEP+K1(1)*K2(1)*(DEFF/DX+K3(1))
    C(1)=K1(1)*K2(1)*(-DEFF/DX+K3(1))
    D1(1)=X1*CK1(1)*CCOAL(1)*STEP+Y4(1)
    DO 43 I=2,NDX
      A(I)=-K1(I)*K2(I-1)*(DEFF/DX+K4(I))
      B1(I)=1.0D0+K1(I)*K2(I)*(DEFF/DX+K3(I))-K1(I)*K2(I-1)
    &      *(-DEFF/DX+K4(I))+(CK2(I)+CK3(I))*STEP
      C(I)=K1(I)*K2(I)*(-DEFF/DX+K3(I))
43    D1(I)=X1*CK1(I)*CCOAL(I)*STEP+Y4(I)
    A(NDXP)=-K1(NDXP)*K2(NDX)*(DEFF/DX+K4(NDXP))
    B1(NDXP)=1.0D0+K1(NDXP)*(KG*K2(NDXP)-K2(NDX)*(-DEFF/DX+K4(NDXP)))
    &      +(CK2(NDXP)+CK3(NDXP))*STEP+K1(NDXP)*K2(NDXP)*K3(NDXP)
    C(NDXP)=0.0D0
    D1(NDXP)=X1*CK1(NDXP)*CCOAL(NDXP)*STEP+Y4(NDXP)+K1(NDXP)
    &      *K2(NDXP)*WVB*KG
C

```

```

      CALL TRINI (A,B1,C,D1,Y4,NDXP)
C
C   RATES FOR TEMPERATURE EQUATIONS
C
      DO 436 I=1,NDXP
      R1(I)=X1*CK1(I)*CCOAL(I)-(CK2(I)+CK3(I))*Y4(I)
      R2(I)=CK2(I)*Y4(I)
436   R3(I)=0.000
C
C
      B2(1)=1.000+K1(1)*K2(1)*(DEFF/DX+K3(1))
      B3(1)=B2(1)
      D2(1)=CK2(1)*Y4(1)*STEP+Y5(1)
      D3(1)=Y6(1)
      DO 435 I=2,NDX
      B2(I)=1.000+K1(I)*K2(I)*(DEFF/DX+K3(I))-K1(I)*K2(I-1)
      &      *(-DEFF/DX+K4(I))
      B3(I)=B2(I)
      D2(I)=CK2(I)*Y4(I)*STEP+Y5(I)
435   D3(I)=Y6(I)
      B2(NDXP)=1.000+K1(NDXP)*(KG+K2(NDXP)-K2(NDX)*(-DEFF/DX+K4(NDXP)))
      &      +K1(NDXP)*K2(NDXP)*K3(NDXP)
      B3(NDXP)=B2(NDXP)
      D2(NDXP)=CK2(NDXP)*Y4(NDXP)*STEP+Y5(NDXP)+K1(NDXP)*K2(NDXP)*
      &      WGB*KG
      D3(NDXP)=Y6(NDXP)+K1(NDXP)*K2(NDXP)*WIB*KG
C
      CALL TRINI (A,B2,C,D2,Y5,NDXP)
      CALL TRINI (A,B3,C,D3,Y6,NDXP)
C
      DO 44 I=1,NDX
      Y1(I)=-DEFF*(Y4(I+1)-Y4(I))/DX+(Y4(I+1)+Y4(I))*K3(I)
      Y2(I)=-DEFF*(Y5(I+1)-Y5(I))/DX+(Y5(I+1)+Y5(I))*K3(I)
44   Y3(I)=-DEFF*(Y6(I+1)-Y6(I))/DX+(Y6(I+1)+Y6(I))*K3(I)
      Y1(NDXP)=KG*(Y4(NDXP)-WVB)+Y4(NDXP)*K3(NDXP)
      Y2(NDXP)=KG*(Y5(NDXP)-WGB)+Y5(NDXP)*K3(NDXP)
      Y3(NDXP)=KG*(Y6(NDXP)-WIB)+Y6(NDXP)*K3(NDXP)
C
C   TEMPERATURE
C
      DO 441 I=1,NDXP
      K6(I)=Y1(I)*CPTAR/2.0+Y2(I)*CPGAS/2.000+Y3(I)*CPINERT/2.000
      K6(I)=K6(I)*K2(I)*STEP
      K7(I)=(R1(I)+R2(I))*CPTAR
      K7(I)=K7(I)*K5(I)

```



```

      K8(I)=Y1(I)*HFTARV+Y2(I)*HFGAS+Y3(I)*HFINERT
      K8(I)=K8(I)*K2(I)*STEP
      K9(I)=(R1(I)+R2(I))*HFTARL
      K9(I)=K9(I)*K5(I)
441    K10(I)=COALK*K2(I)*STEP/DX
      A4(1)=0.0D0
      B4(1)=CP*K5(1)/STEP+K6(1)+K10(1)-K7(1)
      C4(1)=K6(1)-K10(1)
      D4(1)=CP*K5(1)/STEP*Y7(1)-K8(1)+K9(1)
      DO 442 I=2,NDX
        A4(I)=-K6(I-1)-K10(I-1)
        B4(I)=CP*K5(I)/STEP+K6(I)-K6(I-1)-K7(I)+K10(I)+K10(I-1)
        C4(I)=K6(I)-K10(I)
442    D4(I)=CP*K5(I)/STEP*Y7(I)-K8(I)+K8(I-1)+K9(I)
      A4(NDXP)=-K6(NDX)-K10(NDX)
      B4(NDXP)=CP*K5(NDXP)/STEP+2.0D0*K6(NDXP)-K6(NDX)-K7(NDXP)+
&          K10(NDX)+HOV*K2(NDXP)*STEP
      C4(NDXP)=0.0D0
      D4(NDXP)=CP*K5(NDXP)/STEP*Y7(NDXP)-K8(NDXP)+K8(NDX)+K9(NDXP)
&          +HOV*TEMP*STEP*K2(NDXP)
C
      CALL TRINI(A4,B4,C4,D4,Y7,NDXP)
C
      WLW=R1(1)*(R(1))**3.0D0/(DP**3.0D0/8.0D0)
      WLG=R2(1)*(R(1))**3.0D0/(DP**3.0D0/8.0D0)
      DO 991 I=2,NDXP
        WLWV=R1(I)*(R(I))**3.0D0-R(I-1)**3.0D0/(DP**3.0D0/8.0D0)
        WLGV=R2(I)*(R(I))**3.0D0-R(I-1)**3.0D0/(DP**3.0D0/8.0D0)
        WLW=WLW+WLWV
991    WLG=WLG+WLGV
      37  TWLV=TWLV+WLW
      66  TWLG=TWLG+WLG
      TWL=(TWLG+TWLV)*STEP
      WVS=WLW*DP/6.0*DP/DEML/SHN+WVB
      WGS=WLG*DP/6.0*DP/DEML/SHN+WGB
      WLR=WLW+WLG
      IF(TEMP-Y7(1)-1.0D0)88,88,3
      3  CONTINUE
      88  TTWLV=STEP*TWLV
      TTWLG=STEP*TWLG
      DVTIME = T
      IF ( DEB ) WRITE (6,10) T,TTWLV,TTWLG,TWL
      RETURN
      10  FORMAT(//,4X,'TIME FOR PYROLYSIS = ',F10.3, ' SEC',
&          */,4X,'WEIGHT LOSS OF TAR = ',F10.4, ' FRACTION OF ORIGINAL WEIGHT

```

\*OF COAL',  
\*/.4X,'WEIGHT LOSS OF GAS = ',F10.4,' FRACTION OF ORIGINAL WEIGHT  
\*OF COAL',  
\*/.4X,'TOTAL WEIGHT LOSS OF COAL = ',F10.4, ' FRACTION OF ORIGINAL  
\*WEIGHT OF COAL')  
END

## TRIDIAGONAL MATRIX SOLVER SUBROUTINE

```
SUBROUTINE TRINI(A,B,C,D,Y,N)
SAVE
REAL*8 A(N),B(N),C(N),D(N),Y(N),BETA(50),GAMA(50)
BETA(1)=B(1)
GAMA(1)=D(1)/BETA(1)
DO 10 I=2,N
    BETA(I)=B(I)-A(I)*C(I-1)/BETA(I-1)
10    GAMA(I)=(D(I)-A(I)*GAMA(I-1))/BETA(I)
    Y(N)=GAMA(N)
DO 20 I=2,N
    M=N-I+1
20    Y(M)=GAMA(M)-C(M)*Y(M+1)/BETA(M)
RETURN
END
```

## HEAT TRANSFER COEFFICIENT SUBROUTINE

C  
C  
C

CONVECTION HEAT TRANSFER COEFFICIENT SUBROUTINE

```
SUBROUTINE CONV( TEMPC, DP, HC)
  IMPLICIT REAL*8 (A-H,O-Z)
  COMMON / HE / H(9), CP(9)
  COMMON / FF / F(8), AREA
  DIMENSION XK(9), PR(9), XM(9)
  COMMON / PPP / FN2
  DATA PR / 0.0,0.0,.71,.71,.74,.66,.70,1.0,.7 /
  DATA XM / 0.0,0.0,28.,44.,32.,2.,16.,18.,28. /
  XK(3)=4.3+0.0635*TEMPC
  XK(4)=-19.62+.0934*TEMPC
  XK(5)=5.6+.0636*TEMPC
  XK(6)=51.6+.397*TEMPC
  XK(7)=-21.17+.193*TEMPC
  XK(8)=-7.0+.0828*TEMPC
  XK(9)=6.7+.0575*TEMPC
  XKAV=FN2*XK(9)
  XPRAV=FN2*PR(9)
  SUMG=FN2
  SUMM=FN2*XM(9)
  CPAV=FN2*CP(9)
  DO 10 I=3,8
    SUMG = SUMG + F(I)
    SUMM = SUMM + F(I)*XM(I)
    CPAV = CPAV + F(I)*CP(I)
    XKAV = XKAV + F(I)*XK(I)
10   PRAV = PRAV + F(I)*PR(I)
  CPAV=CPAV/SUMM
  XKAV=XKAV/SUMG/418680.0D0
  PRAV=PRAV/SUMG
  XMUAV=PRAV*XKAV/CPAV
  RE=SUMM*DP/AREA/XMUAV
  XNU=2.06*RE**(.425)*PRAV**(1/3)/.4
  HC=XNU*XKAV/DP
  RETURN
  END
```

## HEAT RECOVERY CYCLE SUBROUTINE

```
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C   HEAT RECOVERY SUBROUTINE
C
C   POINT   LOCATION
C     1     SATURATED WATER LEAVING THE CONDENSER
C     2     COMPRESSED WATER LEAVING PUMP #1
C     3     COMPRESSED WATER LEAVING THE FEEDWATER HEATER
C     4     COMPRESSED WATER LEAVING PUMP #2
C     5     SATURATED STEAM LEAVING HP ECONOMIZER
C     6     SUPERHEATED STEAM LEAVING THE SUPERHEATER
C     7     SUPERHEATED STEAM ENTERING TURBINE #1 (PIPE LOSS CORRECT)
C     8     SUPERHEATED STEAM ENTERING TURBINE #2
C     9     SATURATED STEAM LEAVING TURBINE #2
C    10     SATURATED STEAM ENTERING TURBINE #3
C    11     SATURATED STEAM ENTERING TURBINE #4
C    12     SATURATED ENTERING FEEDWATER HEATER
C    13     SATURATED STEAM ENTERING THE CONDENSER
C    14     COMPRESSED WATER LEAVING PUMP #3
C    15     SATURATED STEAM LEAVING LP ECONOMIZER
C    16     SUPERHEATED STEAM MIXING WITH STEAM LEAVING TURBINE #2
C
C   VARIABLES:
C
C     COUNTA2 = COUNTS THE NUMBER OF ITERATIONS FOR CALCULATING TA2
C     COUNTA3 = COUNTS THE NUMBER OF ITERATIONS FOR CALCULATING TA3
C     COUNTA4 = COUNTS THE NUMBER OF ITERATIONS FOR CALCULATING TA4
C     CPAIR   = THE CP FOR THE PRODUCTS ENTERING THE HEATEXCHANGERS
C     CPPROD  = THE CP FOR THE PRODUCTS ENTERING THE HEATEXCHANGERS
C     CPWATER = THE CP FOR WATER
C     CPWATER1 = THE CP FOR WATER IN THE HIGH PRESSURE ECONIMIZER
C     CPWATER2 = THE CP FOR WATER IN THE LOW PRESSURE ECONIMIZER
C     DIFFA2   = THE DIFFERENCES BETWEEN TO VALUES FOR TA2
C     DTMEAN   = THE LOG MEAN TEMPERATURE DIFFERENCE IN THE SUPERHEAT
C     H#       = THE ENTHALPY AT THE ABOVE APPLICABLE POINTS
C     HF#      = THE LIQUID ENTHALPY AT THE ABOVE APPLICABLE POINTS
C     HG#      = THE GASEOUS ENTHALPY AT THE ABOVE APPLICABLE POINTS
C     MDOAIR   = THE MASS FLOW RATE OF GAS THROUGH THE HEATEXCHANGER
C     MDOBT1  = THE MASS FLOW RATE OF WATER THROUGH THE HP ECONIMIZER
C     MDOBT2  = THE MASS FLOW RATE OF WATER THROUGH THE LP ECONIMIZER
C     MDOTF   = THE MASS FLOW RATE OF WATER/STEAM TO FEEDWATER HEATER
C     MDOGTAS = THE MASS FLOW RATE OF STEAM TO THE GASIFIER
```

C NP = THE EFFICIENCY OF THE PUMPS  
C NT = THE EFFICIENCY OF THE TURBINES  
C P# = THE PRESSURE AT EACH OF THE ABOVE POINTS  
C QBD = HEAT LOSS THROUGH STEAM BLOWDOWN  
C QCONDENSER = HEAT REJECTED OUT OF THE CONDENSER  
C QGAS = HEAT LOSS WHEN STEAM GOES TO THE GASIFIER  
C QHPBOIL = HEAT TRANSFERED TO THE WATER IN THE HP ECONIMIZER  
C QIN = HEAT ADDED TO THE SYSTEM FROM THE PRODUCT GAS  
C QLPBOIL = HEAT TRANSFERED TO THE WATER IN THE LP ECONIMIZER  
C QPIPELOSS = HEAT LOSS DO PIPE LOSSES  
C QSUPER = HEAT TRANSFERED TO THE STEAM IN THE SUPERHEATER  
C RCRIT1 = CRITICAL VALUE USED TO HELP DETERMINE C1  
C RCRIT2 = CRITICAL VALUE USED TO HELP DETERMINE C1  
C RCRIT3 = CRITICAL VALUE USED TO HELP DETERMINE C2  
C RCRIT4 = CRITICAL VALUE USED TO HELP DETERMINE C2  
C RSTAR1 = EITHER RCRIT1 OR RCRIT2 WHICH IS GREATER  
C RSTAR2 = EITHER RCRIT3 OR RCRIT4 WHICH IS GREATER  
C R1 = THE INVERSE OF C1(1)  
C R2 = THE INVERSE OF C1(2)  
C R3 = THE INVERSE OF C2(1)  
C R4 = THE INVERSE OF C2(2)  
C S# = THE ENTROPY AT THE ABOVE APPLICABLE POINTS  
C SF# = THE LIQUID ENTROPY AT THE ABOVE APPLICABLE POINTS  
C SG# = THE GASEOUS ENTROPY AT THE ABOVE APPLICABLE POINTS  
C T# = THE TEMPERATURE AT EACH OF THE ABOVE POINTS  
C TA1 = THE TEMPERATURE OF THE GAS ENTERING THE SUPERHEATER  
C TA2 = THE TEMPERATURE OF THE GAS ENTERING THE HP BOILER  
C TA2PRIME = THE TEST VALUE FOR TA2  
C TA3 = THE TEMPERATURE OF THE GAS ENTERING THE LP BOILER  
C TA4 = THE TEMPERATURE OF THE GAS LEAVING THE LP BOILER  
C TESTA2 = THE ACCURACY CHECK FOR TA2  
C TESTA3 = THE ACCURACY CHECK FOR TA3  
C TESTA4 = THE ACCURACY CHECK FOR TA4  
C T12T = TRIAL VALUE FOR THE TEMPERATURE AT POINT 12  
C T16T = TRIAL VALUE FOR THE TEMPERATURE AT POINT 16  
C UABOIL1 = THE HEAT TRANSFER COEFICIENT FOR ECONIMIZER #1  
C UABOIL2 = THE HEAT TRANSFER COEFICIENT FOR ECONIMIZER #2  
C UASUPER = THE HEAT TRANSFER COEFICIENT FOR THE SUPERHEATER  
C UA1 = THE UA FOR ECONIMIZER #1 AT A TRIAL TEMPERATURE  
C UA2 = THE UA FOR ECONIMIZER #2 AT A TRIAL TEMPERATURE  
C V# = THE VOLUVE AT THE ABOVE APPLICABLE POINTS  
C VF# = THE LIQUID VOLUME AT THE ABOVE APPLICABLE POINTS  
C VG# = THE GASEOUS VOLUME AT THE ABOVE APPLICABLE POINTS  
C WNET = THE NET WORK OF THE HEATRECOVERY SYSTEM  
C WPUMPR1 = THE WORK OF HEATRECOVERY PUMP #1

C WPUMPR2 = THE WORK OF HEATRECOVERY PUMP #2  
 C WPUMPR3 = THE WORK OF HEATRECOVERY PUMP #3  
 C WPUMP1 = THE WORK OF HEATRECOVERY PUMP #1 PER KILOGRAM  
 C WPUMP2 = THE WORK OF HEATRECOVERY PUMP #2 PER KILOGRAM  
 C WPUMP3 = THE WORK OF HEATRECOVERY PUMP #3 PER KILOGRAM  
 C WTURBR1 = THE WORK OF HEATRECOVERY PUMP #1  
 C WTURBR2 = THE WORK OF HEATRECOVERY PUMP #2  
 C WTURBR3 = THE WORK OF HEATRECOVERY PUMP #3  
 C WTURBR4 = THE WORK OF HEATRECOVERY PUMP #4  
 C WTURB1 = THE WORK OF HEATRECOVERY PUMP #1 PER KILOGRAM  
 C WTURB2 = THE WORK OF HEATRECOVERY PUMP #2 PER KILOGRAM  
 C WTURB3 = THE WORK OF HEATRECOVERY PUMP #3 PER KILOGRAM  
 C WTURB4 = THE WORK OF HEATRECOVERY PUMP #4 PER KILOGRAM  
 C X# = QUALITY AT THE ABOVE APPLICABLE POINTS

ARRAYS:

C C1(2) = CONSTANTS USED TO DETERMINE UA1  
 C C2(2) = CONSTANTS USED TO DETERMINE UA2  
 C DIFFA3(2) = DIFFERENCES IN TWO TRIAL VALUES OF TA3  
 C DIFFA4(2) = DIFFERENCES IN TWO TRIAL VALUES OF TA4  
 C MDOT(60) = THE MASS FLOW RATES AT EACH POINT OF THE MAIN SYSTEM  
 C P(60) = THE PRESSURES AT EACH POINT OF THE MAIN SYSTEM  
 C T(60) = THE TEMPERATURES AT EACH POINT OF THE MAIN SYSTEM

CC

SUBROUTINE HEATRECOVERY(T,P,MDOT,WNET,WPUMPR1,WPUMPR2,WPUMPR3,  
 & WTURBR1,WTURBR2,WTURBR3,WTURBR4,QSUPER,QHPBOIL,QLPBOIL,  
 & QCONDENSER,QPIPELOSS,CPPROD)  
 REAL\*8 WNET,TA1,TA2,TA3,TA4,P1,P2,P3,P4,P5,P6,P7,P8,P9,P10,P11,  
 & P12,P13,P14,P15,P16,CPAIR,MDOTAIR,NP,NT,MDOTGAS,CPWATER,  
 & CPWATER1,CPWATER2,UABOIL1,UABOIL2,WTURBR1,UA1,WTURBR2,WTURBR3,  
 & WTURBR4,WTURB,WPUMP,WPUMP1,WPUMP2,WPUMP3,C1,C2,DIFFA3,DIFFA4,  
 & RCRIT1,RCRIT2,RCRIT3,RCRIT4,R1,R2,R3,R4,MDOTB1,MDOTB2,MDOTF,  
 & COUNTA2,COUNTA3,COUNTA4,VF1,VG1,HF1,HG1,SF1,SG1,VF3,VG3,HF3,  
 & HG3,SF3,SG3,VF5,VG5,HF5,HG5,SF5,SG5,UA2,RSTAR1,RSTAR2,V6,H6,S6,  
 & V12,H12,S12,VF12,VG12,HF12,HG12,SF12,SG12,V10,H10,S10,VF10,  
 & VG10,HF10,HG10,SF10,SG10,X10,V16,H16,S16,VF16,VG16,HF16,HG16,  
 & SF16,SG16,H2,H4,V7,H7,S7,T12T,T16T,H8,S8,H9,S9,H11,S11,TESTA2,  
 & TESTA3,TESTA4,DIFFA2,UASUPER,DTMEAN,TA2PRIME,VF15,VG15,HF15,  
 & HG15,SF15,SG15,H13,S13,H14,QIN,T1,T2,T3,T4,T5,T6,T7,T8,T9,T10,  
 & T11,T12,T13,T14,T15,T16,MDOTGASE,QPIPELOSS,MDOT,T,P,QREJ,  
 & CPPROD,QCONDENSER,QSUPER,QHPBOIL,QLPBOIL,WTURB1,WTURB2,WTURB3,

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& WTURB4,WPUMPR1,WPUMPR2,WPUMPR3,QGAS,QBD
INTEGER I
DIMENSION C1(2),DIFFA3(2),C2(2),DIFFA4(2),MDOT(60),T(60),P(60)
P1=10.0DO
P2=140.0DO
P3=140.0DO
P4=6276.0DO
P5=P4
P6=P4
P14=1207.0DO
P7=.95DO*P6
P8=2172.0DO
P9=P14*.95DO
P10=P14*.95DO
P11=140.0DO/.95DO
P12=140.0DO
P13=10.0DO
P15=P14
P16=P14*.95DO
TA1=T(19)
T6=838.70DO
T7=T6-3.0DO
NP=.6DO
NT=.75DO
UABOIL1=1082.7DO
UABOIL2=700.0DO
MDOTAIR=.985DO*MDOT(19)
MDOTGAS=MDOT(22)
CPAIR=CPPROD
CPWATER=4.167DO
CALL DSAT(T1,P1,VF1,VG1,HF1,HG1,SF1,SG1,2)
WPUMP1=VF1*(P2-P1)/NP
T2=WPUMP1/CPWATER+T1
H2=WPUMP1+HF1
CALL DSAT(T3,P3,VF3,VG3,HF3,HG3,SF3,SG3,2)
WPUMP2=VF3*(P4-P3)/NP
T4=WPUMP2/CPWATER+T3
H4=WPUMP2+HF3
WPUMP3=VF3*(P14-P3)/NP
T14=WPUMP3/CPWATER+T3
H14=WPUMP3+HF3
TA2=480.0DO+273.15DO
CALL DSAT(T5,P5,VF5,VG5,HF5,HG5,SF5,SG5,2)
CALL DSUPER(T6,P6,V6,H6,S6,1)
COUNTA2=0.0DO

```



```

10  COUNTA2=COUNTA2+1.0D0
    COUNTA3=0.0D0
    CPWATER1=(HF5-H4)/(T5-T4)
    RCRIT1=(HG5-HF5)/CPWATER1/(TA2-T5)
    RCRIT2=CPWATER1*(TA2-T4)/(HG5-HF5+CPWATER1*(T5-T4))
    IF(RCRIT1.GE.RCRIT2)THEN
        RSTAR1=RCRIT1
    ELSE
        RSTAR1=RCRIT2
    ENDIF
    R1=RSTAR1/(1.D0-DEXP(-UABOIL1/MDOTAIR/CPAIR))
    R2=R1+(UABOIL1/MDOTAIR/CPAIR)*(R1-RSTAR1)
    C1(1)=1/R1
    C1(2)=1/R2
20  COUNTA3=COUNTA3+1.0D0
    DO 30 I=1,2
        UA1=MDOTAIR*CPAIR*(C1(I)/(C1(I)-1.0D0)*DLOG((TA2-T5-C1(I))*(HG5-
&      HF5)/CPWATER1)/(TA2-T4-C1(I))*((HG5-HF5)/CPWATER1+T5-T4)))+
&      DLOG((TA2-T5)/(TA2-T5-C1(I))*((HG5-HF5)/CPWATER1)))
30  DIFFA3(I)=UA1-UABOIL1
    TESTA3=DABS(DIFFA3(2))
    C1(2)=(C1(2)*DIFFA3(1)-C1(1)*DIFFA3(2))/(DIFFA3(1)-DIFFA3(2))
    C1(1)=.97*C1(2)
    IF(COUNTA3.GT.20.0D0)GO TO 99
    IF(TESTA3-.1)40,40,20
40  MDOTB1=C1(2)*CPAIR/CPWATER1*MDOTAIR
    TA3=TA2-C1(2)*(T5-T4+(HG5-HF5)/CPWATER1)
    TA2PRIME=TA1-.97D0*MDOTB1*(H6-HG5)/CPAIR/MDOTAIR
    DIFFA2=TA2-TA2PRIME
    TESTA2=DABS(DIFFA2)
    TA2=(TA2+TA2PRIME)/2.0D0
    IF(COUNTA2.GT.90.0D0)GO TO 99
    IF(TESTA2-.1D0)50,50,10
50  DTMEAN=(TA1-T6-TA2+T5)/DLOG((TA1-T6)/(TA2-T5))
    UASUPER=MDOTAIR*CPAIR*(TA1-TA2)/DTMEAN
    COUNTA4=0.0D0
    CALL DSAT(T15,P15,VF15,VG15,HF15,HG15,SF15,SG15,2)
    CPWATER2=(HF15-H14)/(T15-T14)
    RCRIT3=(HG15-HF15)/CPWATER2/(TA3-T15)
    RCRIT4=CPWATER2*(TA3-T14)/(HG15-HF15+CPWATER2*(T15-T14))
    IF(RCRIT3.GE.RCRIT4)THEN
        RSTAR2=RCRIT3
    ELSE
        RSTAR2=RCRIT4
    ENDIF

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R3=RSTAR2/(1.0DO-DEXP(-UABOIL2/CPAIR/MDOTAIR))
R4=R3+(UABOIL2/MDOTAIR/CPAIR)*(R1-RSTAR2)
C2(1)=1/R3
C2(2)=1/R4
60  COUNTA4=COUNTA4+1.0DO
    DO 70 I=1,2
        UA2=MDOTAIR*CPAIR*(C2(I)/(C2(I)-1.0DO)*DLOG((TA3-T15-C2(I)*(HG15
&      -HF15)/CPWATER2)/(TA3-T14-C2(I)*((HG15-HF15)/CPWATER2+T15-
&      T14)))+DLOG((TA3-T15)/(TA3-T15-C2(I)*((HG15-HF15)/
&      CPWATER2))))
70  DIFFA4(I)=UA2-UABOIL2
    TESTA4=DABS(DIFFA4(2))
    C2(2)=(C2(2)*DIFFA4(1)-C2(1)*DIFFA4(2))/(DIFFA4(1)-DIFFA4(2))
    C2(1)=.97DO*C2(2)
    IF(COUNTA4.GT.90.DO)GO TO 98
    IF(TESTA4-.1)80,80,60
80  MDO TB2=C2(2)*CPAIR/CPWATER2*MDOTAIR
    TA4=TA3-C2(2)*(T15-T14+(HG15-HF15)/CPWATER2)
    CALL DSUPER(T7,P7,V7,H7,S7,1)
    CALL DTURBINE(NT,P7,P8,T7,T8,H7,H8,S7,S8,WTURB1)
    CALL DTURBINE(NT,P8,P9,T8,T9,H8,H9,S8,S9,WTURB2)
    CALL DSAT(T16T,P16,VF16,VG16,HF16,HG16,SF16,SG16,2)
    T16=T15-3.0DO
    IF(T16.GT.T16T)THEN
        CALL DSUPER(T16,P16,V16,H16,S16,1)
    ELSE
        CALL DSAT(T16,P16,VF16,VG16,HF16,HG16,SF16,SG16,2)
        H16=HG16
        S16=SG16
    END IF
    H10=((.97DO*MDOTB1-MDOTGAS)*H9+.97DO*MDOTB2*H16)/(.97DO*MDOTB1+
& .97DO*MDOTB2-MDOTGAS)
    CALL DSAT(T10,P10,VF10,VG10,HF10,HG10,SF10,SG10,2)
    IF(H10.GT.HG10)THEN
        CALL DSUPER(T10,P10,V10,H10,S10,3)
    ELSE
        I10=(H10-HF10)/(HG10-HF10)
        S10=I10*SG10+(1.0DO-I10)*SF10
    END IF
    CALL DTURBINE(NT,P10,P11,T10,T11,H10,H11,S10,S11,WTURB3)
    CALL DTURBINE(NT,P11,P13,T11,T13,H11,H13,S11,S13,WTURB4)
    CALL DSAT(T12T,P12,VF12,VG12,HF12,HG12,SF12,SG12,2)
    IF(T12.GT.T12T)THEN
        CALL DSUPER(T12,P12,V12,H12,S12,1)
    ELSE

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```

H12=HG12
S12=SG12
T12=T12T
END IF
MDOTF=(MDOTB1+MDOTB2)*(HF3-H2)/(H12-H2)
WTURB=.97D0*MDOTB1*WTURB1+(.97D0*MDOTB1-MDOTGAS)*WTURB2+(.97D0*
& MDOTB1+.97D0*MDOTB2-MDOTGAS)*WTURB3+(.97D0*MDOTB1+.97D0*MDOTB2-
& MDOTGAS-MDOTF)*WTURB4
WPUMP=(MDOTB1+MDOTB2-MDOTF)*WPUMP1+MDOTB1*WPUMP2+MDOTB2*WPUMP3
WNET=WTURB-WPUMP
MDOT(39)=MDOTF
MDOT(24)=MDOT(18)
MDOT(25)=MDOT(18)
MDOT(26)=MDOT(18)
MDOT(27)=.97D0*MDOTB1+.97D0*MDOTB2-MDOTF-MDOTGAS
MDOT(28)=MDOTB1+MDOTB2-MDOTF
MDOT(29)=MDOT(28)+MDOT(39)
MDOT(30)=MDOTB1
MDOT(31)=MDOTB1
MDOT(32)=.97D0*MDOTB1
MDOT(33)=MDOT(32)
MDOT(34)=MDOT(33)
MDOT(35)=MDOT(34)-MDOT(21)
MDOT(36)=MDOT(35)
MDOT(37)=MDOT(36)+.97D0*MDOTB2
MDOT(38)=MDOT(37)
MDOT(40)=MDOT(38)-MDOT(39)
MDOT(41)=MDOT(40)
MDOT(42)=.03D0*MDOTB1+.03D0*MDOTB2+MDOTGAS
MDOT(43)=MDOT(27)+MDOT(42)
MDOT(44)=MDOTB2
MDOT(45)=MDOTB2
MDOT(46)=.97D0*MDOTB2
MDOT(47)=MDOT(46)
MDOT(48)=.03D0*MDOTB1
MDOT(49)=.03D0*MDOTB2
P(27)=P1
P(28)=P2
P(29)=P3
P(30)=P4
P(31)=P5
P(32)=P6
P(33)=P7
P(34)=P8
P(35)=P8

```

P(36)=P9  
P(37)=P10  
P(38)=P11  
P(39)=P12  
P(40)=P11  
P(41)=P13  
P(42)=P1  
P(43)=P1  
P(44)=P14  
P(45)=P15  
P(46)=P15  
P(47)=P16  
P(48)=P5  
P(49)=P15  
P(24)=P(18)  
P(25)=P(18)  
P(26)=P(18)  
T(24)=TA2  
T(25)=TA3  
T(26)=TA4  
T(27)=T1  
T(28)=T2  
T(29)=T3  
T(30)=T4  
T(31)=T5  
T(32)=T6  
T(33)=T7  
T(34)=T8  
T(35)=T8  
T(36)=T9  
T(37)=T10  
T(38)=T11  
T(39)=T12  
T(40)=T11  
T(41)=T13  
T(42)=T1  
T(43)=T1  
T(44)=T14  
T(45)=T15  
T(46)=T15  
T(47)=T16  
T(48)=T5  
T(49)=T15  
WTURBR1=MDOT(33)\*WTURB1  
WTURBR2=MDOT(35)\*WTURB2

```

WTURB3=MDOT(37)*WTURB3
WTURB4=MDOT(40)*WTURB4
WPUMPR1=MDOT(28)*WPUMP1
WPUMPR2=MDOT(30)*WPUMP2
WPUMPR3=MDOT(44)*WPUMP3
QREJ=(.97DO*MDOTB1+.97DO*MDOTB2-MDOTF-MDOTGAS)*H13
& -(.97DO*MDOTB1+.97DO*MDOTB2-MDOTF-MDOTGAS)*HF1
QCONDENSER=QREJ
QBD=.03DO*MDOTB1*(HG5-HF1)+.03DO*MDOTB2*(HG15-HF1)
QGAS=MDOTGAS*(H8-HF1)-MDOTGAS*(H7-H8)
QPIPELOSS=.97DO*MDOTB1*(H6-H7)+.97DO*MDOTB2*(HG15-H16)
QIN=MDOTAIR*CPAIR*(TA1-TA4)
QSUPER=MDOT(24)*(T(18)-T(24))*CPAIR
QHPBOIL=MDOT(25)*(T(24)-T(25))*CPAIR
QLPBOIL=MDOT(26)*(T(25)-T(26))*CPAIR
RETURN
98 WRITE(6,*)'THE TA3 IS WRONG FOR SOME REASON'
WRITE(6,119)TA3
119 FORMAT(1X,'TEMP OF AIR AT BOILER #2 IS ',6X,F10.3,1X,'K')
RETURN
99 WRITE(6,*)'GUESS ANOTHER TA2'
RETURN
END

```

## J.) STEAM TURBINE SUBROUTINE

```
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C      STEAM TURBINE FOR HEAT RECOVERY SYSTEM SUBROUTINE
C      FOR METRIC UNITS AND IN DOUBLE PRECISION
C
C      VARIABLES:
C
C      HENTER = ENTHALPY OF STEAM ENTERING THE TURBINE
C      HEXIT  = ENTHALPY OF STEAM LEAVING THE TURBINE
C      HFX    = ENTHALPY OF SATURATED STEAM (LIQ) AT EXIT PRESSURE
C      HGX    = ENTHALPY OF SATURATED STEAM (GAS) AT EXIT PRESSURE
C      HXS    = ENTHALPY OF STEAM FOR ISENTROPIC TURBINE
C      NT     = EFFICIENCY OF THE TURBINE
C      PENTER = PRESSURE OF STEAM ENTERING THE TURBINE
C      PEXIT  = PRESSURE OF STEAM EXITING THE TURBINE
C      SENTER = ENTROPY OF STEAM ENTERING THE TURBINE
C      SEXIT  = ENTROPY OF STEAM EXITING THE TURBINE
C      SFX    = ENTROPY OF SATURATED STEAM (LIQ) AT EXIT PRESSURE
C      SGX    = ENTROPY OF SATURATED STEAM (GAS) AT EXIT PRESSURE
C      TENTER = TEMPERATURE OF STEAM ENTERING THE TURBINE
C      TEXTIT = TEMPERATURE OF STEAM EXITING THE TURBINE
C      TXS    = ISENTROPIC TURBINE EXIT TEMPERATURE
C      VEXIT  = THE SPECIFIC VOLUME OF STEAM EXITING THE TURBINE
C      VFX    = THE SPECIFIC VOLUME OF SATURATED STEAM (LIQ) AT
C              EXIT PRESSURE
C      VGX    = THE SPECIFIC VOLUME OF SATURATED STEAM (GAS) AT
C              EXIT PRESSURE
C      WORKTURB = THE TURBINE WORK PER UNIT MASS
C      XR      = THE REAL QUALITY OF STEAM EXITING THE TURBINE
C      XS      = THE ISENTROPIC QUALITY OF STEAM EXITING THE TURBINE
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C      SUBROUTINE DTURBINE(NT,PENTER,PEXIT,TENTER,TEXTIT,HENTER,HEXIT,
&SENER,SEXIT,WORKTURB)
      REAL*8 NT,PENTER,PEXIT,TENTER,TEXTIT,HENTER,HEXIT,SENER,SEXIT,
& WORKTURB,XS,XR,VFX,VGX,HFX,HGX,SFX,SGX,TXS,VIS,HXS,VEXIT
      CALL DSAT(TEXTIT,PEXIT,VFX,VGX,HFX,HGX,SFX,SGX,2)
      IF(SENER.GE.SGX)THEN
        CALL DSUPER(TXS,PEXIT,VIS,HXS,SENER,2)
        WORKTURB=NT*(HENTER-HXS)

```

```
HEXIT=HENTER-WORKTURB
CALL DSUPER(TEXIT,PEXIT,VEXIT,HEXIT,SEXIT,3)
ELSE
  XS=(SENER-SFX)/(SGI-SFX)
  HIS=XS*HGI+(1.0DO-XS)*HFX
  WORKTURB=NT*(HENTER-HIS)
  HEXIT=HENTER-WORKTURB
  XR=(HEXIT-HFX)/(HGI-HFX)
  SEXIT=XR*SGI+(1.0DO-XR)*SFX
ENDIF
RETURN
END
```

## K.) STEAM TABLES SUBROUTINES

```

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C      SUBROUTINE SUPER CALCULATES THE PROPERTIES OF STEAM GIVEN
C      EITHER TEMPERATURE AND PRESSURE OR PRESSURE AND ENTHALPY OR
C      PRESSURE AND ENTROPY.
C
C      N = 1   GIVEN TEMPERATURE AND PRESSURE
C      N = 2   GIVEN PRESSURE AND ENTHALPY
C      N = 3   GIVEN PRESSURE AND ENTROPY
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
      SUBROUTINE SUPER (T,P,V,H,S,N)
      REAL*8 T,P,V,S,H,PROP,X,DIFF,PR,TR,B,C,XI,RHO,RHOZ,TEST
      INTEGER N,COUNT,I
      DIMENSION PROP(5),X(2),DIFF(2)
      PR=P/22089.
      IF(N-2) 10,20,30
10    TR=T/647.29
      X(1)=RHOZ(TR,PR)
      X(2)=.98*X(1)
      COUNT=0
15    COUNT=COUNT+1
      DO 16 I=1,2
      CALL SUPERPROP(T,X(I),PROP)
16    DIFF(I)=PROP(3)-P
      X(2)=(X(2)*DIFF(1)-X(1)*DIFF(2))/(DIFF(1)-DIFF(2))
      X(1)=.98*X(2)
      IF(COUNT.GT.10) GO TO 17
      TEST=ABS(DIFF(2)/P)
      IF(TEST.GT..00001) GO TO 15
      H=PROP(4)
      S=PROP(5)
      GO TO 40
17    WRITE(6,600) T,P
600  FORMAT(" DID NOT CONVERGE AT T=",E14.7," P=",E14.7)
      GO TO 40
20    B=8.57-S-.4615*ALOG(P/101.35)
      C=-.2652-.4615*PR
      XI=(-B+SQRT(B*B-.8136*C))/.4086
      TR=XI**.33333
      T=TR*647.29

```



```

RHO=RHOZ(TR,PR)
V=1./RHO
CALL LOOK(T,RHO,P,S,3,5,PROP)
T=PROP(1)
H=PROP(4)
GO TO 40
30 B=3072.3-H
C=-418.2119*PR-130.2
XI=(-B+SQRT(B*B-1210.6*C))/605.3
TR=XI**.4
T=TR+647.29
RHO=RHOZ(TR,PR)
V=1./RHO
CALL LOOK(T,RHO,P,H,3,4,PROP)
T=PROP(1)
S=PROP(5)
40 V=1./PROP(2)
RETURN
END

```

```

C
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```

```

C
C SUBROUTINE LOOK LOOKS FOR SUPER HEATED PROPERTIES GIVEN
C THE DENSITY AND THE TEMPERATURE
C

```

```

CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C

```

```

SUBROUTINE LOOK(X1,X2,Y1,Y2,M,N,PROP)
REAL*4 X1,X2,Y1,Y2,PROP,DX,B1,B2,DX1,TEST,PROPO,DX2,A11,A12,A21,
& A22
INTEGER N,M,COUNT
DIMENSION PROP(5),PROPO(5)
COUNT=0
10 COUNT=COUNT+1
CALL SUPERPROP(X1,X2,PROPO)
B1=Y1-PROPO(M)
B2=Y2-PROPO(N)
DX=.01D0*X1
X1=X1+DX
CALL SUPERPROP(X1,X2,PROP)
A11=(PROP(M)-PROPO(M))/DX
A21=(PROP(N)-PROPO(N))/DX
X1=X1-DX
DX=.01D0*X2
X2=X2+DX

```

```

CALL SUPERPROP(X1,X2,PROP)
A12=(PROP(M)-PROPO(M))/DX
A22=(PROP(N)-PROPO(N))/DX
I2=X2-DI
DX2=(B2-A21*B1/A11)/(A22-A21*A12/A11)
DX1=(B1-A12*DX2)/A11
TEST=ABS(DX2/X2)+ABS(DX1/X1)
I2=X2+DX2
X1=X1+DX1
IF(COUNT.GT.10)GO TO 20
IF(TEST.GT..00001) GO TO 10
GO TO 30
20 WRITE(6,600)X1,X2
600 FORMAT(' NO CONVERGE, T=',E14.7,' RHO=',E14.7)
30 CALL SUPERPROP(X1,X2,PROP)
RETURN
END

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C FUNCTION RHOZ
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
FUNCTION RHOZ(TR,PR)
REAL*4 TR,PR,Z,X,RHOZ
IF(TR-1.0)20,10,10
10 X=(PR/TR**3.0)**2.0
Z=1.0-X*(1.289/TR-.532*TR*X)
GO TO 30
20 Z=1.0-.7*(PR/TR)**2.0
30 RHOZ=73.94*PR/Z/TR
RETURN
END

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C SUBROUTINE SUPERPROP CALCULATES THE SUPERHEATED STEAM PROPERTIES
C GIVEN THE TEMPERATURE AND DENSITY OF THE STEAM
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
SUBROUTINE SUPERPROP(T,RHO,PROP)
REAL*4 T,RHO,PROP,A,RHOA,C,G,CP,H,HI,E,TA,XM,TC,PC,RHOC,TO,R,S,
& U,TAU,TAUC,TEMP,P,TE

```

```

INTEGER I,J
DIMENSION PROP(5),A(10,7),RHOA(7),C(7),G(6),CP(7),H(7),HI(7)
DATA(A(I,1),I=1,10)/2.9492937E-2,-1.3213917E-4,2.7464632E-7,
*-3.6093828E-10,3.4218431E-13,-2.4450042E-16,1.5518535E-19,
*5.9728487E-24,-.41030848,-4.1605860E-4/
DATA(A(I,2),I=1,10)/-5.198586E-3,7.7779182E-6,-3.3301902E-8,
*-1.6254622E-11,-1.7731074E-13,1.2748742E-16,1.3746153E-19,
*1.5597836E-22,.3373118,-2.0988866E-4/
DATA(A(I,3),I=1,10)/6.8335354E-3,-2.6149751E-5,6.532639E-8,
*-2.6181978E-11,0.,0.,0.,0.,-1.3746618E-1,-7.3396848E-4/
DATA(A(I,4),I=1,10)/-1.5641040E-4,-7.2546108E-7,-9.2734289E-9,
*4.3125840E-12,0.,0.,0.,0.,6.7874983E-3,1.0401717E-5/
DATA(A(I,5),I=1,10)/-6.3972405E-3,2.6409282E-5,-4.7740374E-8,
*5.6323130E-11,0.,0.,0.,0.,.13687317,6.4581880E-4/
DATA(A(I,6),I=1,10)/-3.9661401E-3,1.5453061E-5,-2.9142470E-8,
*2.9568796E-11,0.,0.,0.,0.,7.9847970E-2,3.9917570E-4/
DATA(A(I,7),I=1,10)/-6.9048554E-4,2.7407416E-6,-5.1028070E-9,
*3.9636085E-12,0.,0.,0.,0.,1.3041253E-2,7.1531353E-5/
DATA(RHOA(J),J=1,7)/634.,1000.,1000.,1000.,1000.,1000.,1000./
DATA(G(I),I=1,6)/4.6E4,1.011249E3,.83893,-2.19989E-4,
*2.46619E-7,-9.7047E-11/
DATA E,TA,XM,TC,PC,RHOC,TO/.0048,1000.,18.016,647.286,22.089,
*317.,273.16/
R=8314.34/XM
S=6696.5776-G(1)/T+G(2)*ALOG(T/TO)+T*(G(3)+T*(G(4)/2.+T*(G(5)/3.
**+T*(G(6)/4.)))+G(1)/TO-TO*(G(3)+TO*(G(4)/2.+TO*(G(5)/3.+TO*(G(6)
**/4.)))
U=2375020.7+G(1)*ALOG(T/TO)+T*(G(2)+T*(G(3)/2.+T*(G(4)/3.+T*(
**G(5)/4.+T*(G(6)/5.)))-TO*(G(2)+TO*(G(3)/2.+TO*(G(4)/3.+TO*
***(G(5)/4.+TO*(G(6)/5.))))
TAU=TA/T
TAUC=TA/TC
C(1)=R*T
C(2)=C(1)*(TAU-TAUC)
CP(1)=R
TEMP=R*TAU
CP(2)=C(2)/T-TEMP
DO 10 I=3,7
C(I)=C(I-1)*(TAU-2.5)
CP(I)=C(I)/T-TEMP*(TAU-2.5+(I-2)*(TAU-TAUC))
10 TEMP=TEMP*(TAU-2.5)
TE=EXP(-E*RHO)
DO 40 J=1,7
TEMP=RHO-RHOA(J)
H(J)=A(1,J)+A(2,J)*(TEMP+RHO)

```

```

HI(J)=RHO*(A(1,J)+A(2,J)*TEMP)
DO 30 I=3,8
H(J)=H(J)+A(I,J)*TEMP*(RHO-RHOA(J)+(I-1.)*RHO)
TEMP=TEMP*(RHO-RHOA(J))
30 HI(J)=HI(J)+A(I,J)*RHO*TEMP
H(J)=H(J)+(A(9,J)+RHO*(2.*A(10,J)-E*A(9,J)-E*A(10,J)*RHO))*TE
H(J)=H(J)*RHO*RHO
40 HI(J)=HI(J)+RHO*(A(9,J)+A(10,J)*RHO)*TE
P=RHO*R*T
DO 50 J=1,7
P=P+C(J)*H(J)
U=U+(C(J)-T*CP(J))*HI(J)
50 S=S-CP(J)*HI(J)
S=S-R*ALOG(RHO)
PROP(1)=T
PROP(2)=RHO
PROP(3)=P/1000.
PROP(4)=U/1000.+PROP(3)/RHO
PROP(5)=S/1000.
RETURN
END

```

```

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C SUBROUTINE SAT CALCULATES THE SATURATION PROPERTIES FOR STEAM
C FOR A GIVEN TEMPERATURE AND PRESSURE
C

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```

CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C

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```

SUBROUTINE SAT(T,P,VF,VG,HF,HG,SF,SG,N)
REAL*4 T,P,VF,VG,HF,HG,SF,SG,DPDT,RHOL,PO,ERR,HFG
INTEGER N,COUNT
IF(N-1) 10,10,20
10 CALL SATPROP(T,P,DPDT,RHOL)
GO TO 30
20 T=4900./(17.572-ALOG(P))
COUNT=0
25 COUNT=COUNT+1
CALL SATPROP(T,PO,DPDT,RHOL)
T=T+(P-PO)/DPDT
ERR=ABS((P-PO)/P)
IF(COUNT.GT.10) GO TO 26
IF(ERR.GT..00001) GO TO 25
GO TO 30
26 WRITE(6,610) T

```

```

610 FORMAT(" NO CONVERGE AT TSAT=",E14.7)
30 VF=1./RHOL
CALL SUPER(T,P,VG,HG,SG,1)
HFG=T*(VG-VF)*DPDT
HF=HG-HFG
SF=SG-HFG/T
RETURN
END

```

```

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC

```

```

C
C SUBROUTINE SATPROP CALCULATES THE SATURATION PROPERTIES FOR
C STEAM GIVEN THE TEMPERATURE, PRESSURE, CHANGE IN PRESSURE WITH
C RESPECT TO TEMPERATURE, AND THE DENSITY
C

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C

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```

SUBROUTINE SATPROP(T,P,DPDT,RHOL)
REAL*4 D,F,T,P,DPDT,RHOL,A,TP,TC,PC,RHOC,I,Y,DYDX,I3
INTEGER I
DIMENSION D(8),F(8)
DATA (D(I),I=1,8)/3.6711257,-28.512396,222.6524,-882.43852,
*2000.276,-2612.2557,1829.7674,-533.5052/
DATA (F(I),I=1,8)/-7.41924,.29721,-.1155286,8.685635E-3,
* 1.094098E-3,-4.39993E-3,2.520658E-3,-5.218684E-4/
DATA A,TP,TC,PC,RHOC/.01,338.15,647.286,22089.,317./
I=A*(T-TP)
Y=F(1)+I*(F(2)+I*(F(3)+I*(F(4)+I*(F(5)+I*(F(6)+I*(F(7)+
*I*(F(8)))))))
DYDX=F(2)+I*(2.*F(3)+I*(3.*F(4)+I*(4.*F(5)+I*(5.*F(6)+
*I*(6.*F(7)+I*(7.*F(8))))))
P=PC*EXP((TC/T-1.)*Y)
DPDT=P*(-Y*TC/T/T+(TC/T-1.)*DYDX*A)
I=(1.-T/TC)**(1./3.)
I3=I*I*I
RHOL=RHOC*(1.+I*(D(1)+I3*(D(4)+D(7)*I3)+I*(D(2)+I3*(D(5)+
*D(8)*I3)+I*(D(3)+D(6)*I3)))
RETURN
END

```

Development of Standardized Air-Blown Coal Gasifier  
Gas Turbine Concepts for Future Electric Power Systems  
Volume II

Appendix A: Fixed Bed Gasifier and Sulfur Sorbent  
Regeneration Subsystem Computer Model Development

**END**

**DATE  
FILMED**

**3 / 25 / 92**

