

A major purpose of the Technical Information Center is to provide the broadest dissemination possible of information contained in DOE's Research and Development Reports to business, industry, the academic community, and federal, state and local governments.

Although portions of this report are not reproducible, it is being made available in microfiche to facilitate the availability of those parts of the document which are legible.

3

COW-860612--6

RECEIVED BY OSTI DR-1836-8
JUN 06 1986

Los Alamos National Laboratory is operated by the University of California for the United States Department of Energy under contract W-7405-ENG-24

LA-UR--86-1567

DE86 011262

TITLE TRANSIENT PERFORMANCE INVESTIGATION OF A SPACE POWER SYSTEM HEAT PIPE

NOTICE

AUTHORS Michael A. Merrigan
Edward S. Keddy
J. Tom Sena

THIS REPORT IS ILLEGIBLE TO A DEGREE THAT PRECLUDES SATISFACTORY REPRODUCTION

SUBMITTED TO American Institute of Aeronautics and Astronautics
AIAA/ASME 4th Joint Thermophysics & Heat Transfer Conference
Boston, Massachusetts
June 2-4, 1986

DISCLAIMER

This report was prepared as an account of work sponsored by an agency of the United States Government. Neither the United States Government nor any agency thereof, nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, product, or process disclosed, or represents that its use would not infringe privately owned rights. Reference herein to any specific commercial product, process, or service by trade name, trademark, manufacturer, or otherwise does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States Government or any agency thereof. The views and opinions of authors expressed herein do not necessarily state or reflect those of the United States Government or any agency thereof.

IN ACCEPTANCE OF THIS NOTICE THE PUBLISHER RECOGNIZES THAT THE U.S. GOVERNMENT HOLDS A NONEXCLUSIVE, EXCLUSIVE, AND/OR OTHER RIGHTS TO PUBLISH OR REPRODUCE THIS DOCUMENT IN WHOLE OR IN PART FOR U.S. GOVERNMENT PURPOSES.

LOS ALAMOS NATIONAL LABORATORY REQUESTS THAT THE PUBLISHER IDENTIFY THIS NOTICE AS WORK PERFORMED UNDER THE AUSPICES OF THE U.S. DEPARTMENT OF ENERGY.

Los Alamos Los Alamos National Laboratory
Los Alamos, New Mexico 87545

MASTER

U.S. GOVERNMENT PRINTING OFFICE: 1986

DISTRIBUTION OF THIS DOCUMENT IS UNLIMITED

TRANSIENT PERFORMANCE INVESTIGATION OF A SPACE POWER SYSTEM HEAT PIPE

by

M. A. Merrigan, E. S. Kedd, and J. T. Sena
Los Alamos National Laboratory
Los Alamos, New Mexico 87545

Abstract

Start-up, shut-down, and peak power tests have been conducted with a molybdenum-lithium heat pipe at temperatures to 1500 K. The heat pipe was radiation coupled to a water cooled calorimeter for the tests with rf induction heating used for the input to the evaporator region. Maximum power throughput in the tests was 36.8 kw corresponding to a power density of 23 kw/cm² for the 1.4 cm diameter vapor space of the annular wick heat pipe. The corresponding evaporator flux density was approximately 150 w/cm² over an evaporator length of 40 cm at peak power. Condenser length for the tests was approximately 3.0 m. A variable geometry radiation shield was used to vary the load on the heat pipe during the tests. Results of the tests showed that liquid depletion in the evaporator region of the heat pipe could occur in shut-down and prevent restart of the heat pipe. Changes in surface emissivity of the heat pipe condenser surface were shown to affect the shut-down and re-start limits.

I. Introduction

Many of the high temperature space power system designs under current consideration incorporate high temperature heat pipes for primary heat transport, thermal conditioning, and heat rejection. The operating characteristics of typical high temperature, high-performance heat pipes for these applications have been the subject of a continuing experimental investigations at Los Alamos National Laboratory.^{1,2,3,4} Recent developments in this ongoing program include the performance characterization of a 4-meter long lithium heat pipe. This heat pipe has been tested

in transient and steady-state operation at temperatures to 1500 K. Tests conducted include start-up from the solid state under load, shut-down from high temperature operation with continuous radiation loading, and high power steady-state operation. The intent of these tests was to resolve questions pertaining to the operation and performance characterization of high length to diameter ratio (L/D) heat pipes under conditions of high radiation loading. Start-up times and power handling capability, shut-down behavior with radiation sink temperatures below the freezing point of the working fluid, and accuracy of presently used performance models were specific questions that were addressed in the test program.

II. Description of Test Hardware

The test heat pipe was fabricated from 1.9-cm outside diameter, low carbon arc cast molybdenum tubing having a wall thickness of 0.15 cm. End closures were machined from bar stock of the same material. A cross section of the heat pipe is shown in Fig. 1. Overall length of the heat pipe was 4.0 meters.

The screen wick was fabricated from a single piece of 400-mesh, plain square weave wire cloth woven from 0.025-mm diameter molybdenum-41 w/o rhenium wire. In fabricating the wick the screen was wrapped on a low carbon steel tube mandrel to form a structure 7.25 layers thick. A series of spot welds along the leading and trailing edge of the wrap were used to hold the edges in place during forming and processing. A sheath, also of low carbon steel, was fitted over the screen layers and the assembly drawn through a sizing die to compress the layers. The reduction in the screen cross section was approximately 20%. The

TEST HEAT PIPE

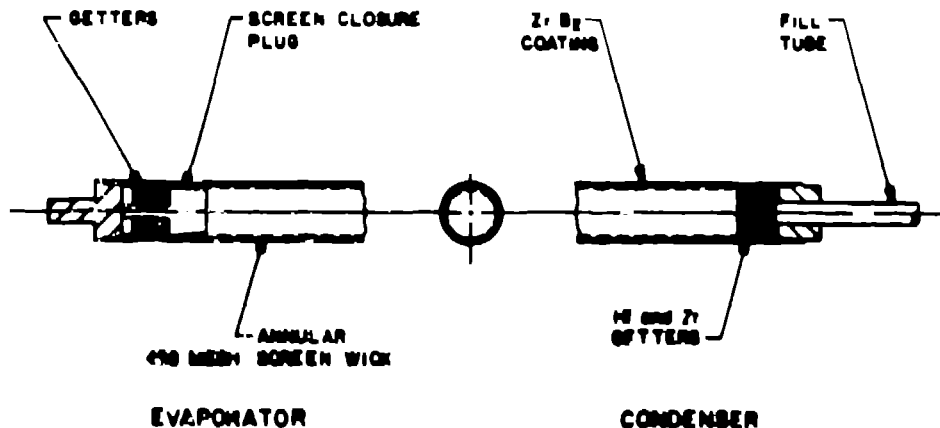


Figure 1. Cross section of the test heat pipe.

steel mandrel and sheath were then removed by dissolution with hydrochloric acid to leave a clean cylindrical wick. The wick was further treated by vacuum firing at 1700 K for 2 hours in order to sinter the screen layers into a monolithic structure and to remove volatile contaminants. Surface tension measurements of the completed wick indicated an effective pore diameter of less than 53 microns. The fluid annulus, which extended the length of the heat pipe, was formed between the container tube inside diameter and the slightly smaller tubular wick. This annulus served as a path for fluid return from the condenser to the evaporator during operation. A molybdenum plug was used to close the screen wick tube at the evaporator end. This plug closure ensured that the annulus pumping capability was established by the effective pore size of the screen wick rather than by the annulus dimensions. The wick outside diameter of 14.9 mm established a maximum dimension for the annulus of 1.16 mm when assembled in the 16.0-mm inside diameter container tube and in line contact with one surface. The cross sectional flow area of the resulting annulus was 28.0 mm². Vapor flow area for the heat pipe, based on the screen wick tube inside diameter was 1.6 cm².

The evaporator end of the screen wick was sealed with a molybdenum plug using a vacuum braze joint made with preplaced zirconium wire braze material. Two turns of 0.381-mm diameter zirconium wire were pressed into the wick. An interference fit of 0.051 mm was maintained between the plug and the bore of the screen wick tube. Brazing was accomplished by rf induction heating the joint in a quartz vacuum envelope that allowed visual observation of the braze material. In making the joint the rf power was slowly increased until the braze was observed to flow. The power and temperature were then immediately reduced and the joint brought to ambient temperature over a period of approximately one-half hour.

Hafnium and zirconium disks were placed inside each end of the heat pipe to serve as getters for carbon, nitrogen, and other trace contaminants during the operation. Closure of the heat pipe envelope was accomplished by electron beam welding a fill plug assembly in place after wick insertion. The fill plug assembly consisted of the tube closure pre-assembled to a fill tube of 6.35-mm molybdenum tubing with a niobium braze transition to stainless steel for fill system attachment. After completion of the weld the system was leak checked with a 9×10^{-11} atm cm³/s leak detector. No leaks were disclosed.

Filling and Processing of the Heat Pipe

The lithium working fluid was placed in the heat pipe by distillation, using the apparatus shown schematically in Fig 2. The inner surfaces of the distillation pot were covered with multiple layers of 100 mesh stainless steel screen to ensure uniform temperatures and to increase the evaporation surface area. The distillation container was heated by rf induction at a level near the surface of the liquid lithium pool to minimize boiling or slugging of the liquid. Hafnium and zirconium getter materials were placed in the distillation pot prior to filling with the lithium charge. The getter materials were placed so as

to remain below the surface of the lithium pool during distillation. A fill volume container was inserted in the fill line between the distillation pot and the heat pipe to establish the fill charge for the heat pipe. Water cooled chill blocks were used as valves on the fill lines during the fill operation.

The internal volume of the heat pipe was connected to a vacuum pumping system through the distillation lines as a means of degassing the system prior to distillation. This same pumping system was used to remove off gases from the lithium during the fill process with a liquid trap providing for return of condensed lithium vapor to the distillation container.

During the vacuum bakeout and distillation processes the heat pipe exterior was protected from oxidation by a quartz envelope connected to a second vacuum system. Pressures were maintained below 10⁻⁴ torr throughout the fill process to reduce the possibility of hydrogen contamination, by disassociation of atmospheric moisture on the heat pipe outer surface.

The heat pipe and fill apparatus were heated to 1050 K and held at temperature until the pressure levels internal to the system reached 10⁻⁶ torr. The charge volume was then cooled

DISTILLATION SYSTEM

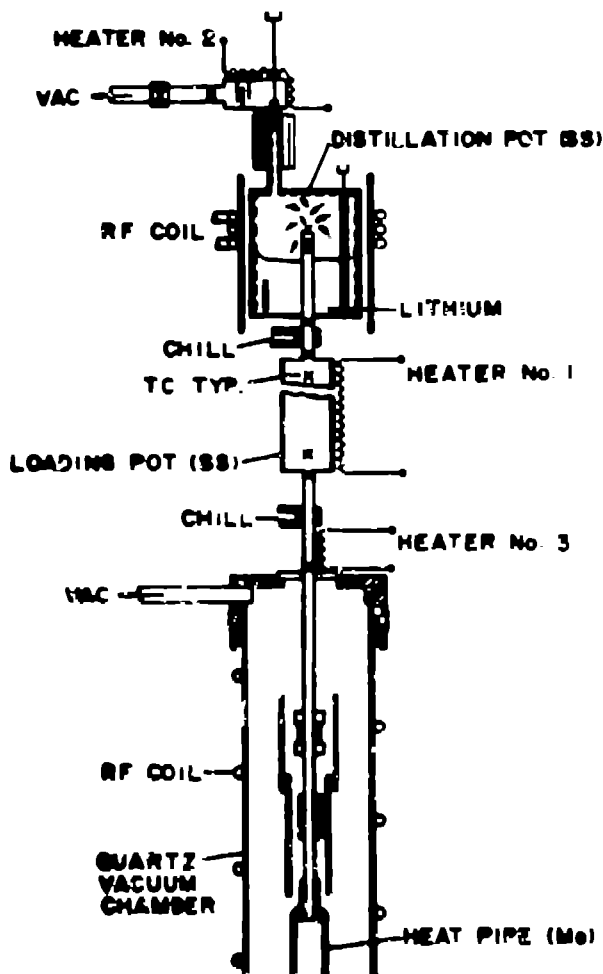


Fig 2 Lithium distillation apparatus used for filling heat pipe

to 500 K, the lower fill line and heat pipe cooled to 300 K and the lower chill block indicated in the figure activated. The distillation pot temperature was then increased to 1150 K and held constant while lithium vapor was condensed in the charge loading pot. When the charge pot had filled it was cooled to 500 K and the lower chill removed. The heat pipe and transfer tube were then heated to 500 K, the loading pot was heated to 800 K and the lithium was allowed to transfer into the heat pipe under the combined gravitational and vapor pressure head. At the end of this process the lower chill block was cooled to close off the heat pipe from the distillation apparatus. The heat pipe was then positioned horizontally for distribution and wet-in of the lithium charge. The heat pipe was heated to 1100 K with a full length rf coil and held at temperature for 5 hours to accomplish the internal surface wetting.

For initial performance verification the heat pipe was heated over a length of 127 cm and the length of the excess liquid pool determined. Final processing of the heat pipe prior to removal of the distillation system and capping consisted of operation under adverse gravity conditions for 4 hours at 1275 K with radiation loading to the laboratory environment.

III. Test Operations

Power input to the heat pipe during the operational tests was provided by high frequency rf induction with a water cooled coaxial calorimeter used as a heat sink for radiation from the condenser as indicated in Fig 3. An adjustable radiation shield was used to vary the loading on the heat pipe during start-up and shut down as shown in Fig 4. Predicted heat pipe performance limits and radiation coupling limits with shutters open and closed are indicated in

Fig. 5. Power throughput was determined from water flow and temperature change measurements at the calorimeter. Heat pipe temperature was measured by tungsten-rhenium thermocouples welded to the heat pipe at the evaporator exit area and at 50 cm intervals along the condenser. Thermocouple locations for the initial tests are summarized in Table I. The tests were conducted in a vacuum chamber maintained at a pressure level of less than 10^{-6} torr throughout the test.

In the initial test the heat pipe was started by heating the evaporator with the radiation shields in the closed position. When the heat pipe reached steady state it was maintained at temperature overnight. The following day the heat pipe was increased in temperature to 1400 K for operational verification. Temperatures along the condenser length, liquid pool length and power throughput were recorded under steady-state operation at 1400 K with the radiation control shutters closed. The power input was then reduced slowly following the radiation coupling curve with the power input turned off once the heat pipe had reached sonic velocity limited operation. After cooling below the lithium solidification temperature over its entire length, the heat pipe was restarted with the chamber radiation shields in the closed position and brought to 1300 K. No operational anomalies were observed during the re-start. After again reaching steady state operating conditions the power input to the heat pipe was turned off instantaneously and the heat pipe again allowed to cool to below the lithium freezing point.

In the next test the radiation shields were opened and the heat pipe was brought to temperature at the same rate as in the previous test. The radiation shields were then closed with the heat pipe at temperature, the power instantaneously shut off and the heat pipe allowed to cool to laboratory ambient conditions. Re-start was accomplished without difficulty.

4m- HEAT PIPE TEST SET-UP

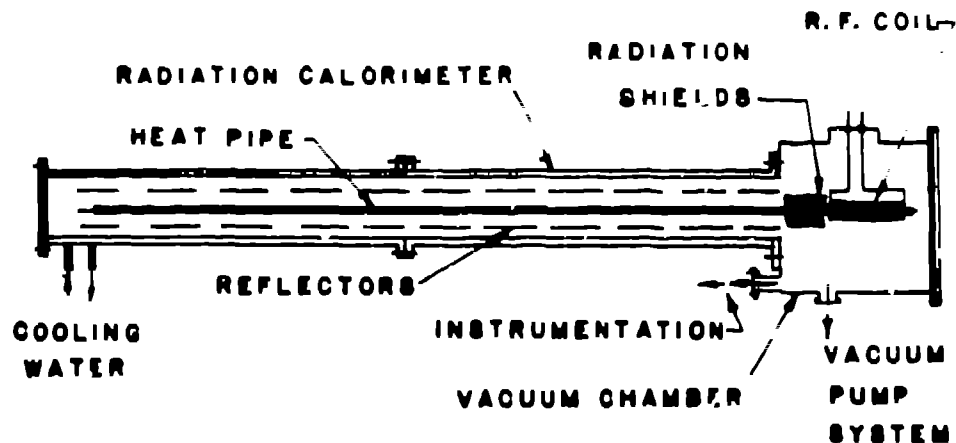


Fig. 3. Heat pipe test set-up

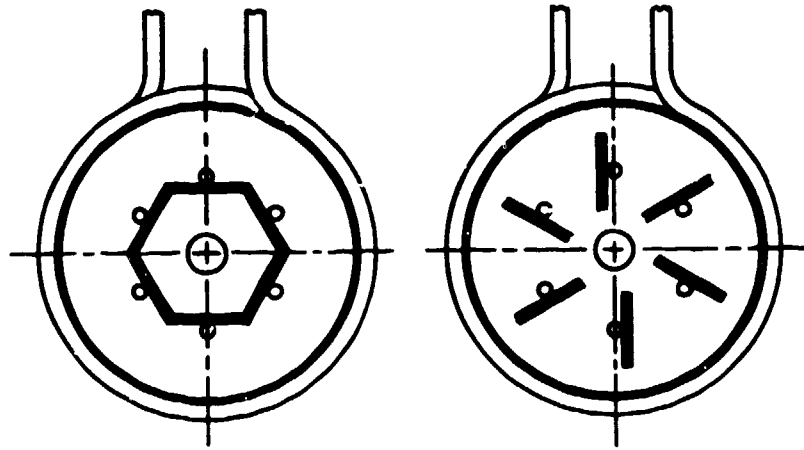


Fig. 4. Adjustable radiation shields used for thermal control.

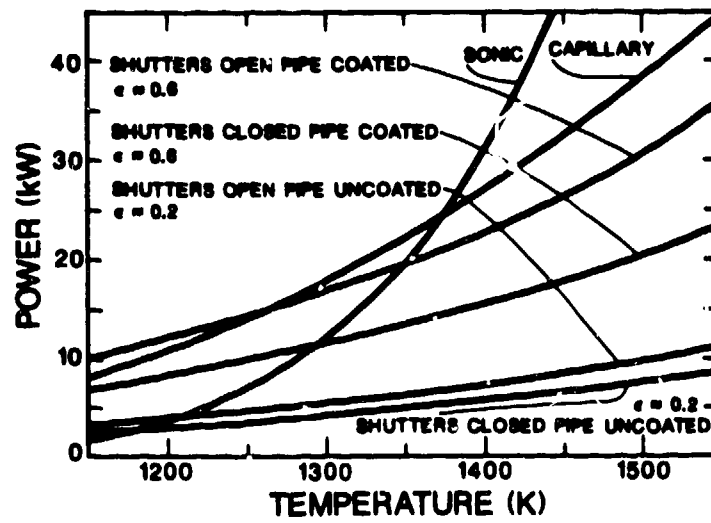


Fig. 5. Heat pipe limits and radiation limits.

TABLE I

Thermocouple Locations Measured from
The Evaporator End of the Heat Pipe

Thermocouple No.	Location (cm)
1	2
2	49
3	57
4	57
5	106
6	163
7	221
8	289
9	347
10	362
11	375
12	375
13	389
14	402

In a succeeding test the heat pipe was taken to a power throughput level of 10.4 kW at 1525 K and shut down under power with the shields open. Following this test the heat pipe was removed from the test chamber for x-ray and neutron radiography to determine the condition of the wick structure and the disposition of the lithium in the annulus. The neutron radiography showed the annulus region of the evaporator to be filled with lithium except for a region approximately 1 cm in diameter located approximately 16.5 cm from the end of the heat pipe. The screen wick was dewetted in this area. The x-ray examination did not show any damage to the screen structure in the dewetted region but did show slight buckling of the screen located about 32 cm from the evaporator end of the heat pipe. The buckling was assumed to be caused by expansion of the evaporator wick during start-up and was not extensive enough to affect heat pipe performance.

The heat pipe was then processed for increased emissivity of the outer surface over all but the evaporator region. This was accomplished by bead blasting the outer surface of the low carbon arc cast molybdenum tube and then DC plasma arc spraying zirconium diboride on the surface. Film thickness of the coating was approximately 0.08 mm with a coating density of about 85% of theoretical. An argon carrier gas was used for the plasma spray process. Emissivity of the resulting coating was approximately 0.6.

The heat pipe was then rewet at 1075 K for 20 hours in a horizontal vacuum furnace. Following the processing it was neutron radiographed and found to be completely wet in. After coating and wet-in the heat pipe was installed in the test chamber, re-instrumented, and operated to verify performance with the radiation shields closed. Peak power throughput during this test was 10.3 kW.

At this point the investigation of shut-down and restart limits was interrupted to perform a high rate start-up test. For this test the radiation shutters in the calorimeter were opened to

the maximum coupling position. At the start of the test the heat pipe was at uniform temperature of approximately 300 K. The evaporator region of the heat pipe was brought to a temperature of 1350 K over a period of approximately 40 minutes and then held constant by increasing the rf input power as the melt front progressed along the length of the heat pipe. Power throughput during the test was determined from flow rate and ΔT measurements for the calorimeter. The resulting temperature history is shown in Fig. 6. A time history of a typical point on the condenser (thermocouple No. 5) is given in Figure 7. Total time to bring the heat pipe to temperature over its 4-m length was approximately 3 hrs with a peak power throughput of 15 kW. At the end of this test the power input was shut off and the heat pipe allowed to cool from temperature with the radiation shutters open. Temperatures along the heat pipe were recorded throughout the cooldown. An axial temperature plot taken some minutes after power shutoff is shown in Fig. 8. The temperature profile shows that the lithium near the evaporator end of the condenser has reached the solidification point while the lithium in the evaporator section of the heat pipe is still almost 240 K above freezing. After complete solidification of the lithium a re-start was attempted with negative results. The heat pipe was again removed from the test chamber and radiographed. The radiograph showed a lithium void in the annulus in the evaporator region. A decision was made at this point to ZrB_2 coat the entire heat pipe, including the evaporator region, in an attempt to provide a more uniform cooldown rate over the length of the heat pipe. It was felt that the formation of shrinkage voids in the evaporator during lithium solidification would be reduced by this technique because of the lower evaporator temperatures at the time that solidification first occurred in the annulus. The heat pipe was therefore reprocessed for full length coating, re-wet, and installed in the test chamber.

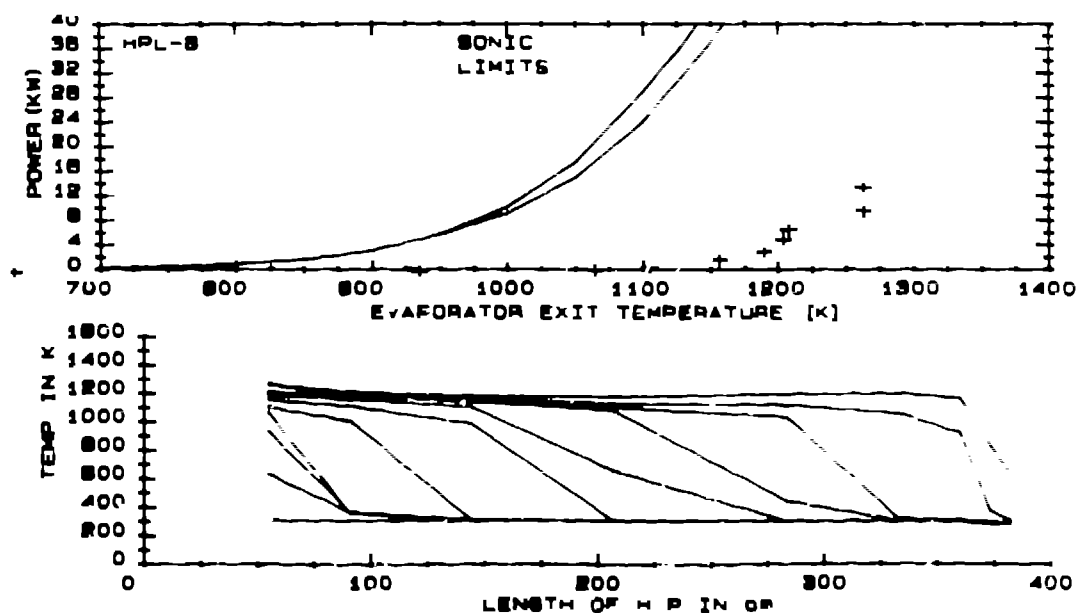


Fig. 6. Heat pipe temperatures versus time during start test

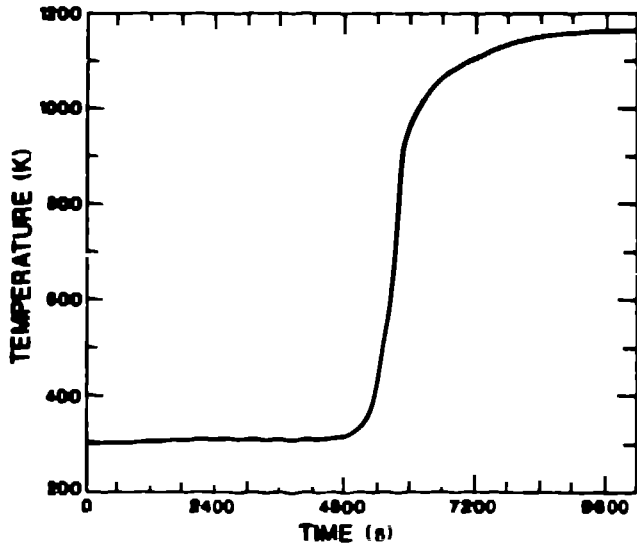


Fig. 7. Time-temperature history for thermocouple No 5.

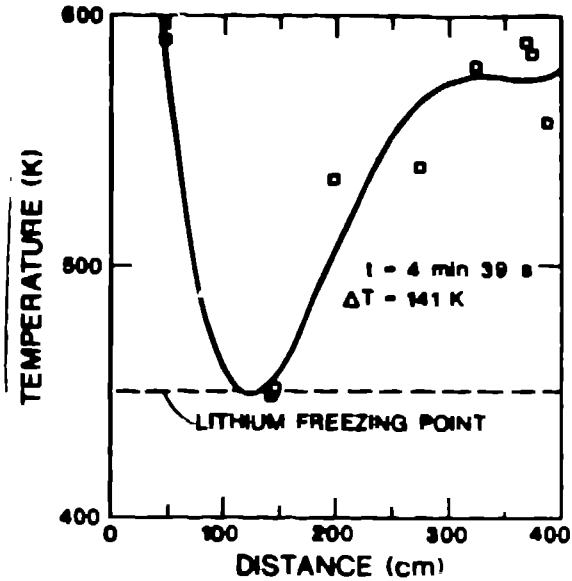


Fig. 9. Heat pipe temperatures during cooldown with evaporator coated

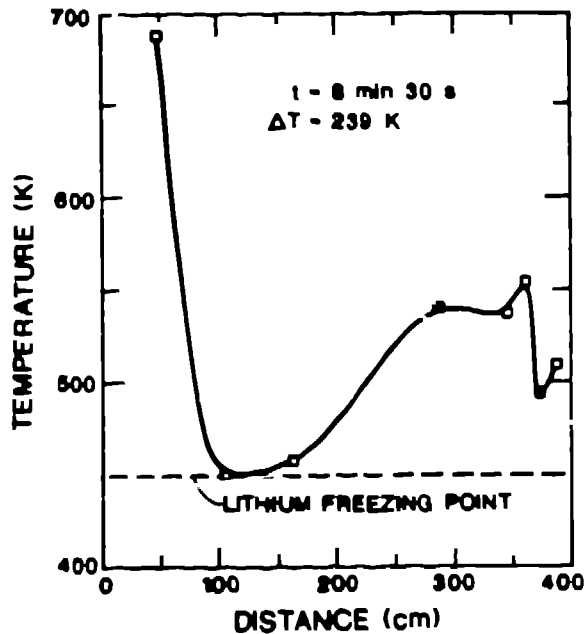


Fig. 8. Heat pipe temperatures during cooldown with uncoated evaporator.

In the following test the heat pipe was brought to 1250 K evaporator exit temperature and 19.04 kW throughput and shut down with the radiation shutters open. Temperature profiles during the shut down are shown in Fig. 9. Comparison with Fig. 8 shows that the difference in temperature between the coolest point at onset of solidification and the evaporator exit temperature has been reduced by approximately 100 K by the use of the high emissivity coating in the evaporator region. The subsequent attempt at restart of the

heat pipe after cooldown to ambient was successful. A summary of the shutdown and re-start tests is given in Table II.

As final performance verification of the heat pipe the radiation shutter apparatus was removed from the test chamber and the heat pipe taken to its power limit with maximum radiation coupling. A temperature-power history for this test is given in Fig. 10. Peak power throughput achieved was 36.8 kW as measured at the calorimeter. Indicated heat pipe evaporator exit temperature at this power level was 1509 K based on corrected optical measurements. The corresponding vapor space power density achieved was 23 kW/cm². A comparison of this demonstrated peak power limit with the capillary limit produced from pre-test measurements of the wick structure using the HPIPE-A model is shown in Fig. 11

IV. Interpretation and Conclusions

Steady-state operation of the heat pipe resulted in temperature profiles in good agreement with the pressure recovery model developed by Busse⁵ and incorporated in the current Los Alamos heat pipe performance prediction code HPIPE-A. As may be seen in Fig. 12 the predicted temperatures are bracketed by experimental data that has a total temperature scatter of less than 2%. The decrease in temperature at the condenser end of the heat pipe indicated in the figure is the result of the surplus liquid pool approximately 40 cm. in length coupled with the use of a water chill at the fill stem at the extreme end of the heat pipe.

Start-up of the radiation loaded heat pipe provided verification of the capability of high power, radiation coupled heat pipes to start under high load conditions, that is, with high emissivity surface coating radiation to an ambient temperature black body calorimeter surface.

TABLE II
Re-Start Test Summary

	PEAK TEMPERATURE (K)	PEAK POWER (kW)	SURFACE CONDITION	SHUT-OFF CONDITION SHIELDS	RESULT
1	1400	5.54	UNCOATED	OPEN	RE-START
2	1300	4.03	UNCOATED	CLOSED	RE-START
3	1361	5.00	UNCOATED	CLOSED	RE-START
4	1525	10.40	UNCOATED	OPEN	NO RE-START
5	1365	10.30	CONDENSER COATED	CLOSED	RE-START
6	1350	15.00	CONDENSER COATED	OPEN	NO RE-START
7	1350	23.3	ALL COATED	OPEN	RE-START
8	1500	30.6	ALL COATED	OPEN	NO RE-START

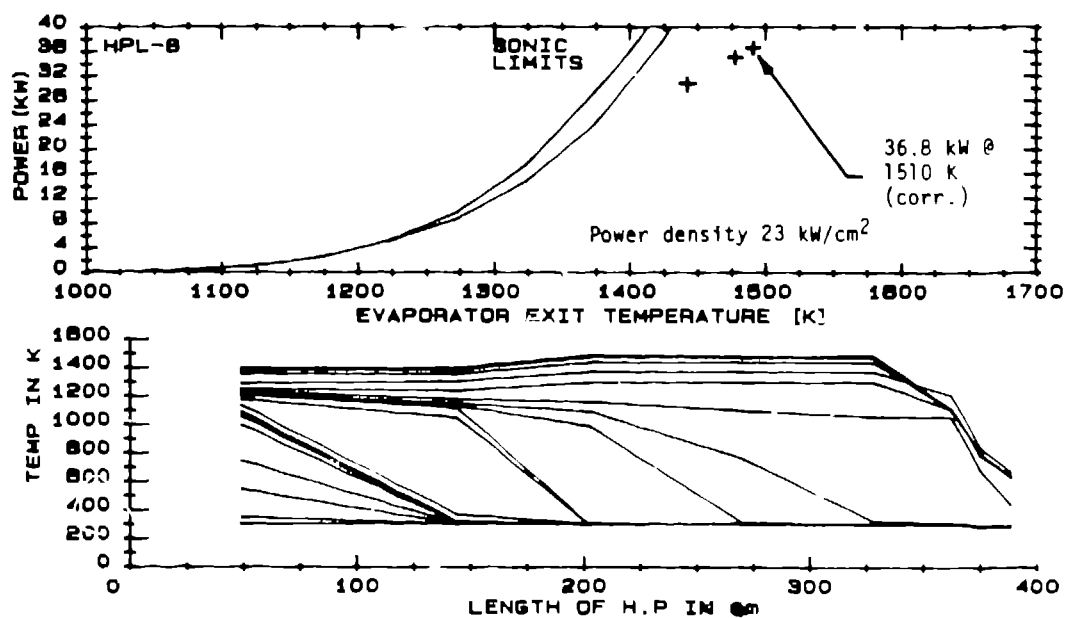


Fig. 10. High power test temperatures and power levels.

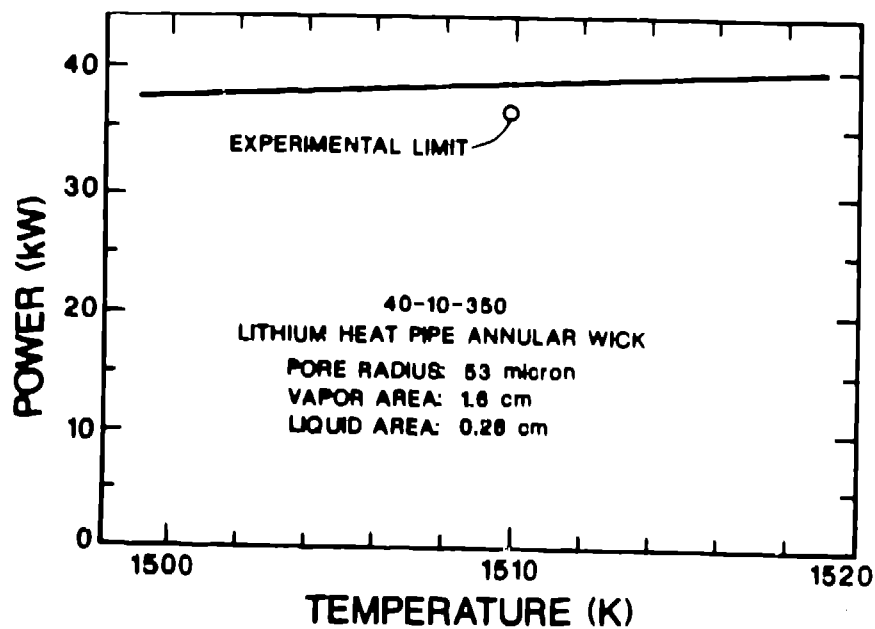


Fig. 11. Comparison of predicted and measured peak power.

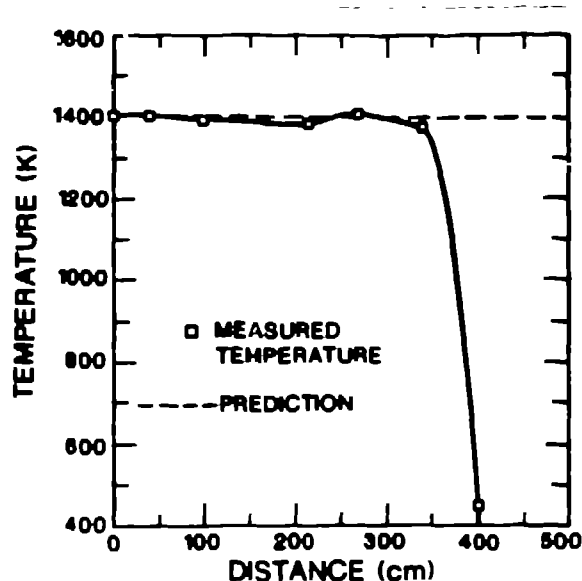


Fig. 12. Comparison of predicted and measured temperature profiles

Shutdown investigations conducted with the 4-m heat pipe were also of interest although the results were not as simply interpreted as those for the start-up case. Primary interest in the shut down investigation was directed to the ability of the heat pipe to restart under normal loads following shutdown and freezing. Test results indicated that restart of the heat pipe without further conditioning following a shut down was dependent on both the radial power density in the condenser, controlled by the surface emissivity of the heat pipe, and on the relative radial power density of the evaporator and condenser regions of the heat pipe during cooldown. It appears that the critical issue is the axial location where freezing first occurs during the shutdown and the ability of the wick structure to accommodate shrinking of the fluid in the region of the heat pipe furthest from the condenser end liquid pool in the period following

the establishment of the freeze plug. The prediction of restart capability for a given radiation coupled heat pipe design will be strongly dependent on the external geometry and thermal environment of a heat pipe as well as on the internal geometry of the heat pipe itself. It is hoped that the heat pipe transient analysis program presently under development by Los Alamos will be capable of this prediction under realistic system conditions.

The high power limit demonstrated in the tests of the 4-m heat pipe has served to advance the boundaries of heat pipe performance by more than 20% and may serve as the basis for the use of heat pipes in more advanced space power system designs.

V. References

1. W. A. Merrigan, J. E. Runyan, H. E. Martinez and E. S. Keddy, "Development and Test of a Space Reactor Core Heat Pipe," AIAA-83-1530, AIAA 18th Thermophysics Conference, June 1983, Montreal, Canada
2. W. A. Merrigan, H. E. Martinez, E. S. Keddy, J. Runyan and J. E. Kemme, "Performance Demonstration of a High Power Space Heat Pipe Design," 18th Intersociety Energy Conversion Engineering Conference, August 1983, Orlando, Florida.
3. W. A. Merrigan, E. S. Keddy, J. R. Runyan, and H. E. Martinez, "Development and Extended Operation of a High Power Radiation Loaded Heat Pipe," 19th AIAA Thermophysics Conference, June 24-28, 1984.
4. E. S. Keddy and H. E. Martinez, "Development of High Temperature Liquid Metal Heat Pipes for Isothermal Irradiation Assemblies," 17th Intersociety Energy Conversion Engineering Conference, August 1982, Los Angeles, California.
5. C. A. Busse, "Pressure Drop in the Vapor Phase of Long Heat Pipes," 1967 Thermionic Conversion Specialist Conference, November 1967