

University of Wollongong

Research Online

Faculty of Engineering and Information
Sciences - Papers: Part B

Faculty of Engineering and Information
Sciences

2019

Characterisation and evaluation of a new phase change enhanced working solution for liquid desiccant cooling systems

Haoshan Ren

University of Wollongong, hr681@uowmail.edu.au

Zhenjun Ma

University of Wollongong, zhenjun@uow.edu.au

Stefan Gschwander

Fraunhofer Institute for Solar Energy Systems ISE

Follow this and additional works at: <https://ro.uow.edu.au/eispapers1>



Part of the [Engineering Commons](#), and the [Science and Technology Studies Commons](#)

Recommended Citation

Ren, Haoshan; Ma, Zhenjun; and Gschwander, Stefan, "Characterisation and evaluation of a new phase change enhanced working solution for liquid desiccant cooling systems" (2019). *Faculty of Engineering and Information Sciences - Papers: Part B*. 2387.

<https://ro.uow.edu.au/eispapers1/2387>

Research Online is the open access institutional repository for the University of Wollongong. For further information contact the UOW Library: research-pubs@uow.edu.au

Characterisation and evaluation of a new phase change enhanced working solution for liquid desiccant cooling systems

Abstract

Desiccant solutions play an essential role in desiccant cooling systems to absorb moisture from the process air. This paper presents the characterisation of a new working solution for liquid desiccant cooling systems. The new working solution was prepared through dispersion of micro-encapsulated phase change materials (MPCMs) into lithium chloride (LiCl) desiccant solutions to ensure that the dehumidification process was achieved under a low temperature condition and to improve thermal capacity and moisture removal efficiency of the mixture. The properties of the new solution, including density, enthalpy-temperature relationship, particle size distribution, thermal conductivity, and vapour pressure were characterised through either experimental tests or theoretical analysis. It was shown that the density and thermal conductivity of the new working solution slightly decreased with the increase of the mass fraction of the MPCMs in the mixture. The thermal capacity of the new working solution substantially increased in the melting temperature range of the MPCMs used. The vapour pressure of the new working solution decreased due to the existence of the MPCM particles. It is expected that the dehumidification efficiency of adiabatic dehumidifiers can be potentially improved when using this new working solution due to the decreased vapour pressure and increased thermal capacity of the phase change enhanced desiccant solution.

Disciplines

Engineering | Science and Technology Studies

Publication Details

Ren, H., Ma, Z. & Gschwander, S. (2019). Characterisation and evaluation of a new phase change enhanced working solution for liquid desiccant cooling systems. *Applied Thermal Engineering*, 150 1197-1205.

1 **Characterisation and evaluation of a new phase change enhanced** 2 **working solution for liquid desiccant cooling systems**

3 Haoshan Ren¹, Zhenjun Ma^{1,*}, Stefan Gschwander²

4 ¹Sustainable Buildings Research Centre, University of Wollongong, 2522, Australia

5 ²Fraunhofer Institute for Solar Energy Systems, Freiburg, 79110, Germany

6 *Email: zhenjun@uow.edu.au

7
8 **Abstract:** Desiccant solutions play an essential role in desiccant cooling systems to absorb
9 moisture from the process air. This paper presents the characterisation of a new working
10 solution for liquid desiccant cooling systems. The new working solution was prepared through
11 dispersion of micro-encapsulated phase change materials (MPCMs) into lithium chloride (LiCl)
12 desiccant solutions to ensure that the dehumidification process was achieved under a low
13 temperature condition and to improve thermal capacity and moisture removal efficiency of the
14 mixture. The properties of the new solution, including density, enthalpy-temperature
15 relationship, particle size distribution, thermal conductivity, and vapour pressure were
16 characterised through either experimental tests or theoretical analysis. It was shown that the
17 density and thermal conductivity of the new working solution slightly decreased with the
18 increase of the mass fraction of the MPCMs in the mixture. The thermal capacity of the new
19 working solution substantially increased in the melting temperature range of the MPCMs used.
20 The vapour pressure of the new working solution decreased due to the existence of the MPCM
21 particles. It is expected that the dehumidification efficiency of adiabatic dehumidifiers can be

22 potentially improved when using this new working solution due to the decreased vapour
23 pressure and increased thermal capacity of the PCM-LiCl desiccant solution.

24 **Keywords:** Desiccant cooling; phase change material; new working solution; characterisation;
25 vapour pressure.

26

27 **Nomenclature**

28 d diameter (m)

29 h enthalpy (J/kg)

30 k thermal conductivity (W/m K)

31 m mass (kg)

32 P vapour pressure (Pa)

33 t time (s)

34 T temperature (°C)

35 x_m mass fraction

36 x_v volume fraction

37

38 *Greek symbols*

39 α coefficient

40 α_R equivalent thermal conductivity depression

41 β coefficient

42 θ reduced temperature

43 π coefficient
44 ρ density (kg/m³)

45

46 *Subscripts*

47 a air

48 c core

49 e equivalent

50 $LiCl$ lithium chloride desiccant solution

51 PCM micro-encapsulated phase change material

52 $PCM-LiCl$ phase change enhanced LiCl desiccant solution

53 s shell

54

55 **1. Introduction**

56 Air conditioning is essential to our lives and is greatly impacting the quality of our life and
57 even saves lives during intense heat waves [1]. The amount of installed air conditioning systems
58 is expected to increase dramatically in the coming decades, largely driven by economic growth
59 and global warming [2]. Nowadays, the majority of air conditioning systems used were
60 developed based on the vapour compression cycle [3]. These systems can control sensible load
61 effectively but are very inefficient to deal with latent load, in particular under hot and humid
62 climatic conditions. This is because a significant amount of energy is required to dehumidify
63 the air by overcooling the air below its dew point temperature in order to remove the moisture

64 through condensation and then heating it to the desired temperature [4].

65 Liquid desiccant cooling as one of the alternative solutions has received increasing attention
66 for air conditioning and dehumidification due to its effectiveness in humidity control and great
67 potentials in energy savings [5-7]. A liquid desiccant is generally a concentrated salt solution
68 such as lithium bromide (LiBr), lithium chloride (LiCl), and calcium chloride (CaCl₂) that
69 directly absorbs moisture without cooling the air below its dew point. Liquid desiccant cooling
70 can be driven by low-grade thermal energy (50-90 °C) such as waste heat or solar energy [8].
71 As the building peak cooling demand generally occurs at approximately the same time as local
72 peak solar radiation, this opens up opportunities of using solar energy to drive liquid desiccant
73 cooling and even to take air conditioning off the grid with the assistance of thermal energy
74 storage systems.

75 Various types of liquid desiccant cooling systems and their potentials to maintain
76 acceptable indoor thermal comfort under different climatic conditions have been studied. Ham
77 et al. [9], for instance, developed a liquid desiccant and dew point evaporative cooling assisted
78 100% outdoor air system. The simulation results showed that 12% of the primary energy can
79 be saved by using this system, in comparison with a typical variable air volume system. Elmer
80 et al. [10] developed a liquid desiccant cooling system consisting of a regenerator, a
81 dehumidifier and an evaporative inter-cooler integrated with an energy exchanger. The
82 experimental results showed that the dehumidification effectiveness of this system was 30-47%
83 with an average electrical COP of 2.5. Chen et al. [11] proposed a liquid desiccant dehumidifier
84 and a regenerative indirect evaporative cooling system for fresh air treatment. The thermal

85 energy obtained from solar collectors was used for liquid desiccant regeneration. The results
86 showed that the energy saving of this system was 22.4-53.2% under various inlet air conditions,
87 in comparison to a conventional chilled water air conditioning system. A solar driven liquid
88 desiccant cooling system was developed by a company named L-DCS Technology [12]. This
89 system was installed in a building in Singapore and used for cooling and dehumidification [13].

90 A dehumidifier is a key component in a liquid desiccant cooling system. Adiabatic
91 dehumidifiers are relatively simple units, but they must work with a high desiccant flow rate
92 and a high air flow rate in order to achieve a better dehumidification efficiency [6, 14]. In
93 adiabatic dehumidifiers, the temperature of the desiccant solution continuously increases along
94 the desiccant flow direction. The increase in the solution temperature deteriorates the
95 dehumidification efficiency as the ability of a desiccant solution to attract water vapour from
96 an air stream decreases with the increase of the solution temperature and the decrease of the
97 solution concentration [15]. In order to ensure dehumidifiers work with high efficiency and
98 minimise the carryover of desiccant droplets, internally cooled dehumidifiers which can allow
99 the desiccant solution working in a low temperature and low flow rate condition were studied
100 [16]. The performance comparison of an adiabatic dehumidifier with an internally cooled
101 dehumidifier showed that the dehumidification effectiveness of the internally cooled
102 dehumidifier was improved from 0.3876 - 0.4771 to 0.4769 - 0.7058, in comparison to that of
103 the adiabatic dehumidifier [17]. However, the inherent complex configuration of the internally
104 cooled dehumidifiers and high maintenance costs make these systems less attractive.

105 Phase change materials (PCMs) with an ability to provide high energy storage densities and

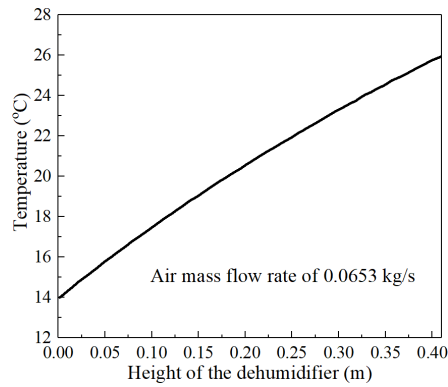
106 the characteristics to store thermal energy at relatively constant temperatures have attracted
107 wide attention for developing high-performance buildings [18, 19]. The development of PCMs
108 and PCM thermal energy storage for liquid desiccant cooling systems have been reported in a
109 few studies [20-22]. Niu et al. [20], for instance, proposed to use micro-nanoencapsulated
110 PCMs as a heat transfer fluid to improve the performance of internally cooled dehumidifiers in
111 liquid desiccant cooling systems. The micro-nanoencapsulated PCMs were prepared and their
112 thermo-physical properties were characterised. However, the real application of such materials
113 in dehumidifiers was not reported. Al-Abidi et al. [21] and Mahdi and Nsofor [22] respectively
114 developed a triplex tube PCM thermal energy storage for liquid desiccant cooling systems. Al-
115 Abidi et al. [21] experimentally tested the charging performance of the PCM thermal storage
116 unit using a paraffin wax with a melting temperature of 82 °C while Mahdi and Nsofor [22]
117 numerically investigated the charging performance of the thermal energy storage using the same
118 paraffin wax but was enhanced by alumina nanoparticles and a porous copper foam. However,
119 both studies did not integrate the PCM thermal energy storage into liquid desiccant cooling
120 systems.

121 In this study, a novel phase change enhanced LiCl (i.e. PCM-LiCl) desiccant solution was
122 proposed to improve the dehumidification efficiency of adiabatic dehumidifiers. The new
123 solution was prepared by dispersing micro-encapsulated PCMs (MPCMs) into LiCl desiccant
124 solutions. As MPCMs have a relatively large thermal storage capacity, the dispersion of
125 MPCMs into liquid desiccant solutions can ensure the mixture work under a low temperature
126 condition, improving overall dehumidification efficiency. The properties of the new PCM-LiCl

127 desiccant solution such as density, enthalpy-temperature ($h-T$) relationship, particle size
128 distribution, thermal conductivity, and vapour pressure were characterised through either
129 experimental measurements or theoretical investigation.

130 **2. Development and characterisation of PCM-LiCl desiccant solutions**

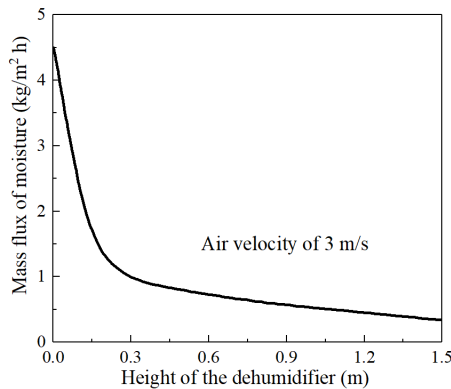
131 Dehumidifiers can generally offer a better performance when the desiccant solution is
132 working in a low temperature condition such as in the range of 20-30 °C [23-24]. As shown in
133 Fig. 1a [25], the temperature of the LiCl desiccant solution is continuously increased along the
134 height of the adiabatic dehumidifier due to the heat and mass transfer between the process air
135 and the liquid desiccant and the absorption heat released during the dehumidification process.
136 This increased temperature will lead to the increase in the vapour pressure of the liquid
137 desiccant and therefore decrease the vapour pressure difference between the process air and
138 liquid desiccant, which deteriorates the moisture transfer and dehumidification effectiveness of
139 the adiabatic dehumidifier. This deterioration can be further confirmed by the results (see Fig.
140 1b) reported in another study [26], in which it was shown that the mass flux of moisture between
141 the process air and liquid desiccant decreased along the flow direction of the liquid desiccant.
142 As PCMs have a large storage density, the dispersion of MPCMs into liquid desiccants can
143 improve the thermal capacity of the mixture and therefore can decrease the temperature increase
144 of the liquid desiccant along the flow direction of the adiabatic dehumidifier, improving overall
145 dehumidification effectiveness.



146

147

a) Temperature of the liquid desiccant [25]



148

149

b) Mass flux of moisture [26]

150

Fig. 1. Temperature of LiCl liquid desiccant and mass flux of moisture along the flow

151

direction of an adiabatic dehumidifier.

152

In this study, two commercially available MPCM products, i.e. BASF Micronal DS 5038X

153

and Micronal DS 5040X with a melting temperature of 25 °C and 23 °C respectively, were used

154

to prepare phase change enhanced LiCl desiccant solutions. Both MPCMs were made of a

155

highly crosslinked polymethylmethacrylate (PMMA) polymer wall and paraffin inside as the

156

PCM. It is noted that the MPCMs used in this study might not be optimal. As the temperature

157

of the inlet working solution of the dehumidifier is a controlled variable, the phase change

158

materials should be selected based on the set-point of the inlet working solution temperature in

159

order to ensure that the PCM undergoes the phase change process during the dehumidification.

160 Deionized water and LiCl with a purity higher than 98% were used to prepare LiCl desiccant
 161 solutions. The PCM-LiCl desiccant solution was prepared through directly dispersing the
 162 MPCM particles into the LiCl desiccant solution via mechanical stirring.

163 2.1 Characterisation of PCM-LiCl desiccant solutions

164 To understand the likely benefits of using this new working solution for air
 165 dehumidification, the properties of the PCM-LiCl desiccant solutions, including density, h - T
 166 relationship, particle size distribution, thermal conductivity, and vapour pressure should be first
 167 characterised. In this study, these properties were characterised through either theoretical
 168 analysis or experimental measurements.

169 2.1.1 Density

170 The density of the PCM-LiCl desiccant solution was determined based on the density of
 171 each insoluble component of the mixture as the potential non-homogeneity of the new solution
 172 might introduce errors in measurements of the mixture. As the PMMA has good resistance to
 173 LiCl and is insoluble in LiCl desiccant solutions, the density of the PCM-LiCl desiccant
 174 solution can be calculated using Eq. (1) [27], in which the density of the MPCM was measured
 175 using a pycnometer (to be introduced in Section 2.2) and the density of the LiCl desiccant
 176 solution was determined using the correlation expressed in Eq. (2) [28].

$$177 \rho_{PCM-LiCl} = \frac{\rho_{PCM}\rho_{LiCl}}{\rho_{PCM}(1-x_{m,PCM}) + \rho_{LiCl}x_{m,PCM}} \quad (1)$$

$$178 \rho_{LiCl} = \rho_{H2O} \sum_{i=0}^3 \alpha_i \left(\frac{x_{m,LiCl}}{1-x_{m,LiCl}} \right)^i \quad (2)$$

179 where ρ is the density, α_i are the parameters and the values of α_0 - α_3 were 1.0, 0.540966, -
 180 0.303792, 0.100791, respectively [28], x_m is the mass fraction, and the subscripts $PCM-LiCl$,

181 *PCM* and *LiCl* indicate phase change enhanced LiCl desiccant solution, micro-encapsulated
 182 phase change material, and LiCl desiccant solution, respectively.

183 2.1.2 Enthalpy-temperature (*h-T*) relationship

184 Differential scanning calorimeter (DSC) tests are commonly used to measure *h-T*
 185 relationships of PCMs. In this study, the *h-T* relationship of the PCM-LiCl desiccant solution
 186 was not measured directly as LiCl desiccant solutions may corrode the metal container of the
 187 DSC device and further damage the equipment. Thus, the *h-T* relationship of the new solution
 188 was determined using Eq. (3), based on the enthalpy of the LiCl desiccant solution determined
 189 using Eq. (4) [29] and the DSC test results of the MPCMs.

$$190 \quad h_{PCM-LiCl} = x_{m,PCM} h_{PCM} + (1 - x_{m,PCM}) h_{LiCl} \quad (3)$$

$$191 \quad h_{LiCl} = A + BT + CT^2 \quad (4)$$

192 where *h* is the enthalpy, *T* is the temperature, and the coefficients of *A*, *B*, and *C* are calculated
 193 using Eqs. (5-7) [29], respectively.

$$194 \quad A = -66.2324 + 11.2711x_{m,LiCl} - 0.79853x_{m,LiCl}^2 + (2.1534E-02)x_{m,LiCl}^3 - (1.66352E-04)x_{m,LiCl}^4$$

$$195 \quad (5)$$

$$196 \quad B = 4.5751 - 0.146924x_{m,LiCl} + (6.307226E-03)x_{m,LiCl}^2 -$$

$$197 \quad (1.38054E-04)x_{m,LiCl}^3 + (1.06690E-06)x_{m,LiCl}^4 \quad (6)$$

$$198 \quad C = (-8.09689E-04) + (2.18145E-04)x_{m,LiCl} - (1.36194E-05)x_{m,LiCl}^2 + (3.20998E-07)x_{m,LiCl}^3 -$$

$$199 \quad (2.64266E-09)x_{m,LiCl}^4 \quad (7)$$

200 2.1.3 Thermal conductivity

201 The thermal conductivity of the PCM-LiCl desiccant solution was calculated using

202 Maxwell's equation as shown in Eq. (8), which has been widely used to calculate the thermal
 203 conductivity of MPCM suspensions [27, 30]. The thermal conductivity of the MPCM particles
 204 (k_{PCM}) was determined using the composite sphere approach reported by Goel et al. [30], in
 205 which the heat transfer resistance of the shell material was determined based on the thickness
 206 of the MPCM shell. The heat transfer resistance of the core material was evaluated based on
 207 the assumption that a solid sphere is in an infinite medium [31]. The thermal conductivity of
 208 the MPCM can then be calculated using Eq. (9) [30]. The diameter of the MPCM particle was
 209 measured using a particle size analyser to be introduced in Section 2.2 and the thickness of the
 210 PMMA shell was determined using the composition of the MPCM measured using a
 211 thermogravimetric analyser (see Section 2.2). The thermal conductivity of the LiCl desiccant
 212 solution is calculated using Eq. (10) [28].

$$213 \frac{k_{PCM-LiCl}}{k_{LiCl}} = \frac{2k_{LiCl} + k_{PCM} + 2x_{v,PCM}(k_{PCM} - k_{LiCl})}{2k_{LiCl} + k_{PCM} - x_{v,PCM}(k_{PCM} - k_{LiCl})} \quad (8)$$

$$214 \frac{1}{k_{PCM}d_{PCM}} = \frac{1}{k_c d_c} + \frac{d_{PCM} - d_c}{k_s d_{PCM} d_c} \quad (9)$$

$$215 k_{LiCl} = k_{H2O} - \alpha_R x_{m,LiCl,e} \quad (10)$$

216 where k is the thermal conductivity, x_v is the volume fraction, d is the diameter, α_R is the
 217 equivalent thermal conductivity depression, $x_{m,LiCl,e}$ is the equivalent ionic concentration, and
 218 the subscripts s and c indicate shell and core, respectively.

219 2.1.4 Vapour pressure

220 The dehumidification performance of a liquid desiccant is directly influenced by the vapour
 221 pressure difference between the liquid desiccant and process air [5]. In this study, the vapour
 222 pressure of the PCM-LiCl desiccant solution was measured using a thermogravimetric method

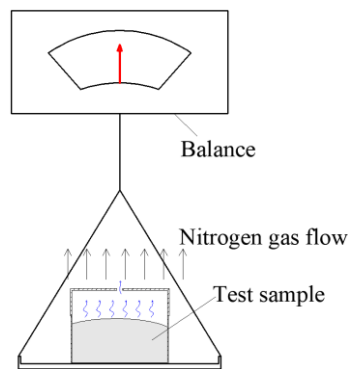
223 [32]. In this method, the mass loss of the sample is measured using a thermogravimetric
224 analyser (i.e. TGA 5500 from TA Instruments was used in this study) and the result is then used
225 to determine the vapour pressure of the solution by using the Langmuir equation for free
226 vaporisation, as shown in Eq. (11) [33].

$$227 \log(P_T) = \beta_1 \log\left(\frac{1}{\beta_2} \left|\frac{dm}{dt}\right|_T\right) + \beta_3 \quad (11)$$

228 where $\left|\frac{dm}{dt}\right|_T$ and P_T are the mass loss rate and the vapour pressure at a given temperature
229 respectively, and β_1 , β_2 and β_3 are the coefficients influenced by the geometry structure of the
230 crucible. In this study, these three coefficients were determined based on the experimental
231 results for a reference substance (i.e. pure water) with the known vapour pressure.

232 During the test, the sample was placed in a sealed crucible with a 0.35 mm-diameter hole
233 on the lid, as illustrated in Fig. 2a. The crucible was then placed on the holder of the balance
234 inside the thermogravimetric analyser (Fig. 2b). The sample was purged by a pure nitrogen gas
235 flow in order to maintain the water content outside the crucible close to zero.

236



237

238

a) Schematic of vapour pressure measurement



b) Thermogravimetric analyser used in this study

Fig. 2. Vapour pressure measurement using a thermogravimetric analyser.

2.2 Characterisation of MPCMs and pure LiCl desiccant solutions

The properties of the MPCMs were determined either using the information provided in the product data sheet [34] or through the experimental measurements.

The MPCM used was in a physical form of powder and the bulk density of the MPCM was provided in the datasheet [34]. However, the particle density of the MPCM was required to determine the density of the PCM-LiCl desiccant solutions. In this study, a pycnometer was used to measure the particle density of the MPCM.

DSC tests were carried out to characterise the h - T relationship of the two MPCMs. The tests were implemented using a micro DSC (micro DSC III, SETARAM) device with a weight of the samples of around 300 mg. The heating/cooling rate used was 0.05 K/min.

The particle size distribution of the MPCM was measured using a laser diffraction particle size analyser and the sample was pre-processed in an ultrasonic bath for deagglomeration of the particles before the measurement. The results were analyzed using Mie model [35], which worked well for homogeneous and spherical particles with a diameter less than 30 μm [36, 37]. The Mie model was derived by solving Maxwell's equations describing electromagnetic

257 radiation for the light scattered by a homogeneous sphere under the uniform illumination [35].

258 Thermogravimetric analysis was carried out using a thermogravimetric analyser to
259 determine the composition of the MPCMs, and then to calculate the thickness of the PMMA
260 shell. During the test, the MPCM was heated to a temperature of 600 °C with a scanning rate of
261 1.0 K/min, and the test was then carried out using a nitrogen gas flow.

262 As LiCl is highly hygroscopic and pure LiCl may absorb the moisture from the air during
263 the preparation. A thermogravimetric analysis test for LiCl desiccant solutions was therefore
264 carried out to determine the mass fraction of LiCl in the solution. During the measurement, the
265 LiCl desiccant solution was heated to a temperature of 350 °C with a scanning rate of 5.0 K/min
266 and was then maintained at 350 °C for 18 hours [38]. The thermogravimetric analysis test was
267 also performed with a nitrogen gas flow.

268 **3. Results of properties characterisation**

269 In this section, the properties of the MPCMs and PCM-LiCl desiccant solutions were
270 respectively characterised using the methods introduced in Section 2. The majority of the tests
271 presented in this study were repeated in order to confirm the consistency of the results.

272 **3.1 Results of characterisation of MPCM properties**

273 The measured densities of the MPCM DS 5038X and MPCM DS 5040X were 990.9 ± 0.6
274 kg/m^3 and $1012.6 \pm 0.4 \text{ kg/m}^3$, respectively. These results were considered to be reasonable as
275 the density of the two major compositions, i.e. paraffin wax and PMMA, were around 850 kg
276 m^{-3} and 1150 kg m^{-3} , respectively.

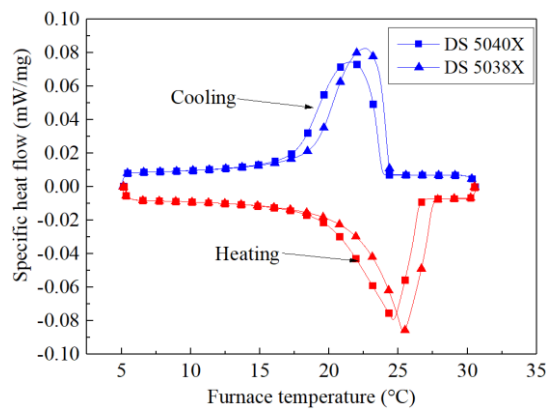
277 The DSC curves of the MPCM DS 5038X and MPCM DS 5040X are presented in Fig. 3

278 and their onset temperatures, peak temperatures, and heat of fusion during the cooling and
 279 heating processes are summarized in Table 1. The heat of fusion of the MPCM was calculated
 280 based on the temperature range of 15-28 °C. It can be observed that the peak phase change
 281 temperatures of the MPCM DS 5038X for both heating and cooling were slightly higher than
 282 those of the MPCM DS 5040X. The results from Table 1 also showed that the onset
 283 temperatures of the MPCM DS 5038X were higher than those of the MPCM DS 5040X during
 284 both cooling and heating processes.

285 **Table 1.** Onset temperatures, peak temperatures, and heat of fusion of the two MPCMs.

	DS 5038X			DS 5040X		
	Onset Temp. (°C)	Peak Temp.(°C)	Heat of fusion (J/g)	Onset Temp. (°C)	Peak Temp. (°C)	Heat of fusion (J/g)
Heating	22.55	25.37	96.7	19.87	24.64	94.8
Cooling	24.37	22.60	96.2	23.83	21.53	93.2

286

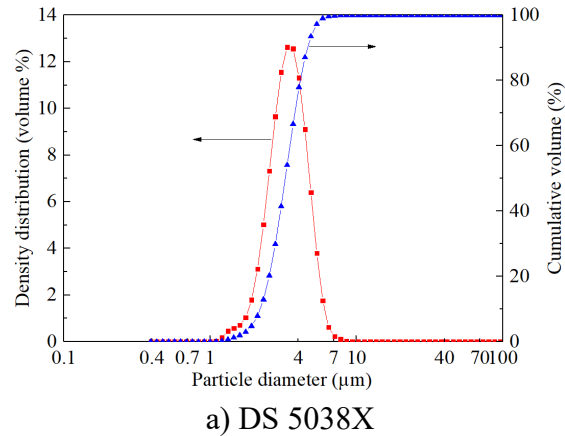


287

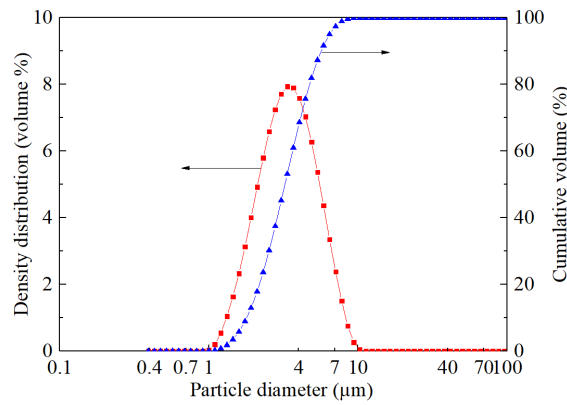
288 **Fig. 3.** DSC test results of DS 5038X and DS 5040X.

289 The measured particle size distributions of the MPCMs are presented in Fig. 4. The
 290 volumetric average diameters of the MPCMs DS 5038X and DS 5040X were determined as
 291 3.51 μm and 3.68 μm , respectively. It can be observed that the particle size distributions of both
 292 MPCMs were in the range of 1 - 10 μm .

293
294



295



296

297 **Fig. 4.** Particle size distribution of the two MPCMs.

298

299

300

301

302

303

304

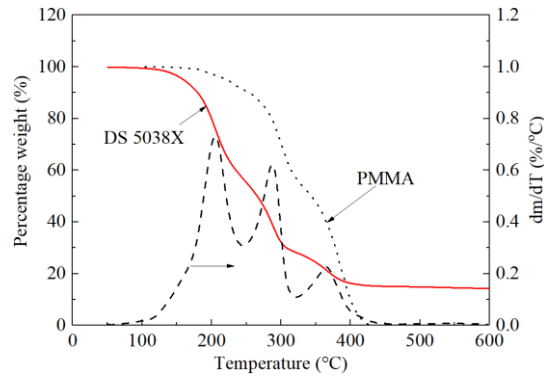
305

306

The results (i.e. percentage weight and mass loss rate of the sample) from the thermogravimetric analysis of the MPCMs DS 5038X and DS 5040X and a sample of the MPCM DS 5038X after the decomposition test are presented in Fig. 5. The thermogravimetric analysis results of the shell material (i.e. PMMA) obtained from [39] were also provided. It can be seen that the thermal degradation of the MPCM DS 5038X can be divided into three phases and the MPCM was mainly decomposed in the first two phases at a temperature below 330 °C, and the PMMA was completely decomposed at a temperature of around 454 °C (Fig. 5a). Similar trends were also observed for the MPCM DS 5040X (Fig. 5b). By comparing the thermogravimetric analysis curve of the MPCM to that of the PMMA, it can be derived that the

307 composition of the MPCM DS 5038X was 17.4% wt. PMMA, 67.5% wt. paraffin, and 15.1%
308 wt. residual and that of the MPCM DS 5040X was 17.1% wt. PMMA, 69.5% wt. paraffin, and
309 13.4% wt. residual. The similar results were also reported in [37, 40].

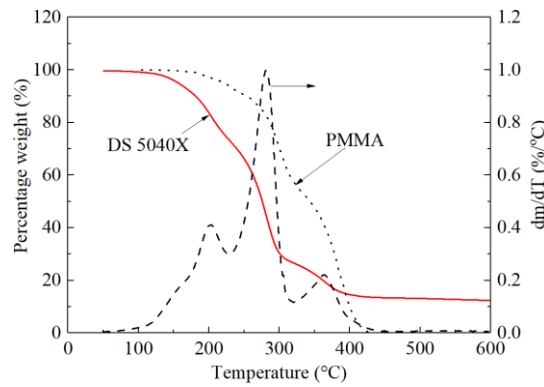
310



311

312

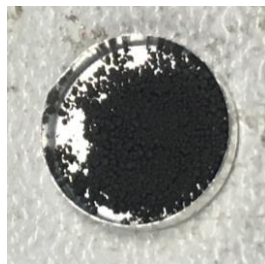
a) DS 5038X



313

314

b) DS 5040X



315

316

c) DS 5038X after the decomposition test

317 **Fig. 5.** Decomposition of PMMA [39], MPCM DS 5038X and MPCM DS 5040X.

318 3.2 Results of characterisation of PCM-LiCl desiccant solutions

319 3.2.1 Verification of vapour pressure measurement

320 The vapour pressure of the LiCl desiccant solution was measured using the

321 thermogravimetric method and the results were compared with the calculated values determined
322 using the correlation expressed in Eq. (12) [28]. The concentration of the LiCl desiccant
323 solution was first determined via a thermogravimetric analysis test and the vapour pressure of
324 the LiCl desiccant solution was then measured at a temperature of 50 °C using the method
325 presented in Fig. 2. The test results are presented in Fig. 6, which were measured based on the
326 initial concentration of the LiCl solution of 29.53%. It can be found that the vapour pressure of
327 the solution decreased with the increase of the solution concentration and the measured values
328 generally agreed well with the calculated results. The average and maximum deviations
329 between the measured and calculated values were 1.9% and 5.5%, respectively. It is worthwhile
330 to note that the total mass of the test sample used was relatively small due to the capacity of the
331 crucible used and the increase of the concentration was resulted by the continuous loss of the
332 water in the solution.

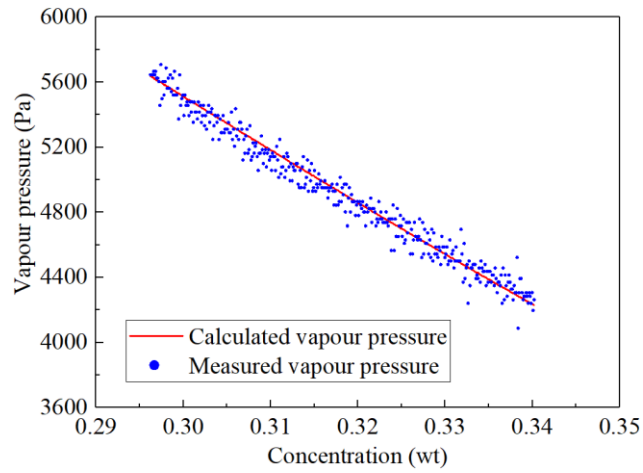
$$333 \quad P_{LiCl} = \pi_{25} f(x_{m,LiCl}, \theta) P_{H2O} \quad (12)$$

334 where θ is the reduced temperature, $f(x_{m,LiCl}, \theta)$ is determined using Eq. (13) [28], and π_{25} is
335 calculated using Eq. (14) [28].

$$336 \quad f(x_{m,LiCl}, \theta) = 2 - \left[1 + \left(\frac{x_{m,LiCl}}{\pi_0} \right)^{\pi_1} \right]^{\pi_2} + \left\{ \left[1 + \left(\frac{x_{m,LiCl}}{\pi_3} \right)^{\pi_4} \right]^{\pi_5} - 1 \right\} \theta \quad (13)$$

$$337 \quad \pi_{25} = 1 - \left[1 + \left(\frac{x_{m,LiCl}}{\pi_6} \right)^{\pi_7} \right]^{\pi_8} - \pi_9 \exp\left(- \frac{(x_{m,LiCl} - 0.1)^2}{0.005} \right) \quad (14)$$

338 where π_0 - π_8 are the coefficients and the values used were provided in Table 2 [28].



339

340 **Fig. 6.** Measured and calculated vapour pressures of the LiCl desiccant solution.

341

342 **Table 2.** Coefficients for vapour pressure calculation of LiCl desiccant solutions [28].

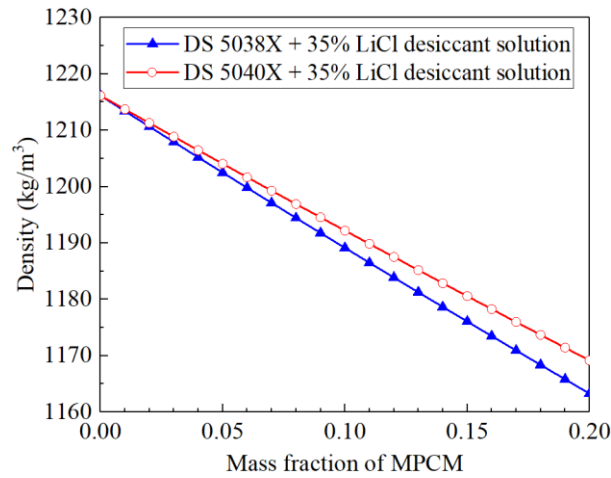
π_0	π_1	π_2	π_3	π_4	π_5	π_6	π_7	π_8	π_9
0.28	4.30	0.60	0.21	5.10	0.49	0.362	-4.75	-0.40	0.03

343

344 3.2.2 Properties characterisation

345 In this study, the LiCl desiccant solution with a concentration of 35% was first prepared
 346 and the MPCMs with different mass fractions were then mixed with the LiCl desiccant solution
 347 to prepare the PCM-LiCl desiccant solutions. The densities of different PCM-LiCl desiccant
 348 solutions were then calculated using Eqs. (1) and (2) and the results are presented in Fig. 7. It
 349 can be seen that the density of the PCM-LiCl desiccant solutions decreased with the increase
 350 of the mass fraction of the MPCMs as the density of the MPCMs was lower than that of the
 351 LiCl desiccant solution. The density of the mixture using the MPCM DS 5038X was always
 352 lower than that using the MPCM DS 5040X which was resulted by the relatively low density
 353 of the MPCM DS 5038X. It is noted that the density of the mixture calculated using Eq. (1) [27]

354 is applicable to homogenous fluids and the PCM-LiCl desiccant solution developed in this study
355 can be considered as homogenous when it is well mixed.



356

357

Fig. 7. Density of the PCM-LiCl desiccant solution.

358

The h - T relationship of the PCM-LiCl desiccant solutions was determined using Eqs. (3)

359

and (4). The heating curve of the MPCM obtained from the DSC measurement was used to

360

evaluate the enthalpy of the MPCM. The results are presented in Fig. 8. The enthalpy of the

361

LiCl desiccant solution without MPCMs was also presented in this figure. It can be observed

362

that the enthalpy of the PCM-LiCl desiccant solution increased with the increase of the solution

363

temperature and the mass fraction of the MPCMs in the mixture. A large increasing rate

364

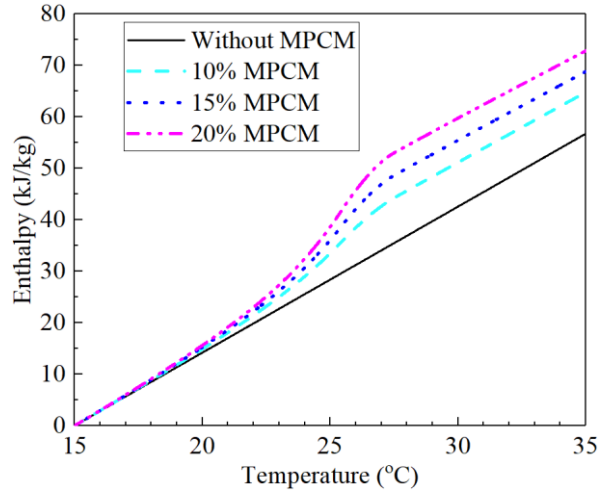
occurred in the MPCM melting temperature range of 20-27 °C. The enthalpy of the solution

365

then increased almost linearly if further increasing the solution temperature. There was not a

366

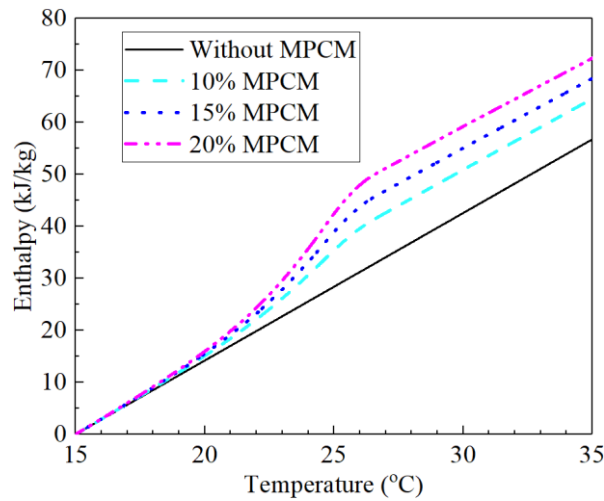
clear difference between the use of two different MPCMs.



367

368

a) DS 5038X



369

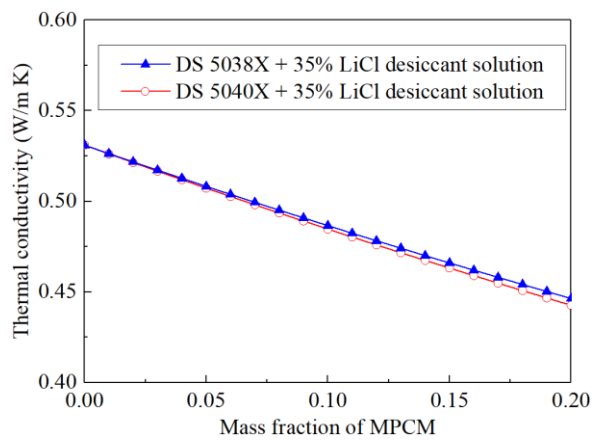
370

b) DS 5040X

371 **Fig. 8.** Enthalpy-temperature relationships of the pure LiCl desiccant solution and PCM-LiCl
 372 desiccant solutions with different mass fractions of the MPCMs.

373 The thermal conductivity of the PCM-LiCl desiccant solution was calculated using Eqs. (8)
 374 and (9) based on the composition and particle size of the MPCM determined. It is noted that
 375 the diameter of the particles dispersed into the LiCl desiccant solution was assumed to be the
 376 same as the volumetric average diameter of the particles. It was also assumed that the residual
 377 of the MPCMs obtained from the thermogravimetric analysis had the same thermal conductivity

378 as the PCM used. The resulted thermal conductivity of the PCM-LiCl desiccant solutions is
379 presented in Fig. 9. It can be observed that the thermal conductivities of the PCM-LiCl desiccant
380 solutions using the MPCM DS 5040X and DS 5038X were very close to each other when the
381 mass fraction of the MPCMs was less than 20%. The thermal conductivity of the PCM-LiCl
382 desiccant solutions decreased with the increase of the mass fraction of the MPCMs as the
383 thermal conductivity of the MPCM particle was lower than that of the LiCl desiccant solution.



384
385 **Fig. 9.** Thermal conductivity of the PCM-LiCl desiccant solutions with different mass
386 fractions of MPCMs.

387 The vapour pressures of the PCM-LiCl desiccant solutions with different mass fractions of
388 the two MPCMs were measured using the thermogravimetric method. As the variation in the
389 vapour pressures of the PCM-LiCl solutions using the two MPCMs showed a similar trend, the
390 results of the PCM-LiCl solution using the MPCM DS 5040X under the temperature of 50 °C
391 were presented only.

392 During the measurement, the LiCl desiccant solution was first prepared and the
393 concentration of the LiCl desiccant solution was determined via a thermogravimetric analysis
394 test. A relatively low initial concentration of the LiCl desiccant solution (i.e. 29.53%) was used

395 to prepare PCM-LiCl desiccant solutions as the concentration of the desiccant solution was
 396 expected to increase during the vapour pressure measurement due to continuous loss of the
 397 water in the solution. The PCM-LiCl desiccant solution was then prepared in the crucible
 398 presented in Fig. 2a. Table 3 summarises the details of the three samples used in the test. It is
 399 worthwhile to note that the mass fraction of the MPCM in each test sample cannot be precisely
 400 controlled during the preparation.

401

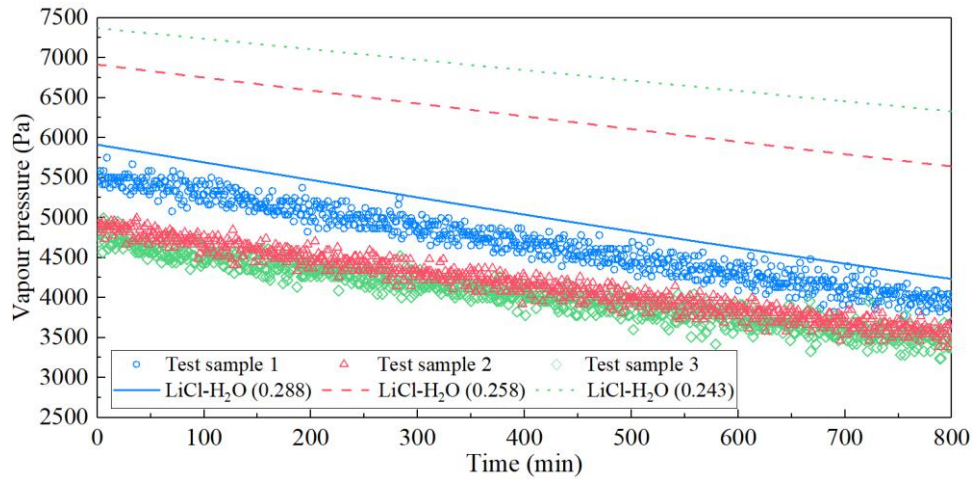
402 **Table 3.** Test samples prepared for vapour pressure measurement.

Test sample	Weight (mg)	Mass fraction of the MPCM	Mass fraction of the LiCl
Sample 1	49.158	0.029	0.288
Sample 2	57.424	0.128	0.258
Sample 3	54.190	0.180	0.243

403

404 The measurement results of the PCM-LiCl desiccant solution using the MPCM DS 5040X
 405 under the temperature of 50 °C are presented in Figs. 10 and 11. The vapour pressures of LiCl
 406 desiccant solutions without the MPCM were also presented, which were calculated based on
 407 the same temperature condition and the same initial mass fractions of LiCl in the desiccant
 408 solutions as those presented in Table 3. The vapour pressure of the pure LiCl desiccant solution
 409 was determined using Eqs. (12)-(14). It can be seen that the vapour pressures of the PCM-LiCl
 410 desiccant solutions of three samples continuously decreased as a function of time. This is
 411 because the mass fractions of both MPCM and LiCl continuously increased with time during
 412 the test due to the water loss (Fig. 11). For each test sample, the vapour pressure of the PCM-
 413 LiCl desiccant solution was always lower than that of the pure LiCl desiccant solution during

414 the test period due to the existence of the MPCM particles. The vapour pressure of the PCM-
 415 LiCl desiccant solution decreased with the increase of the mass fraction of the MPCM in the
 416 mixture.

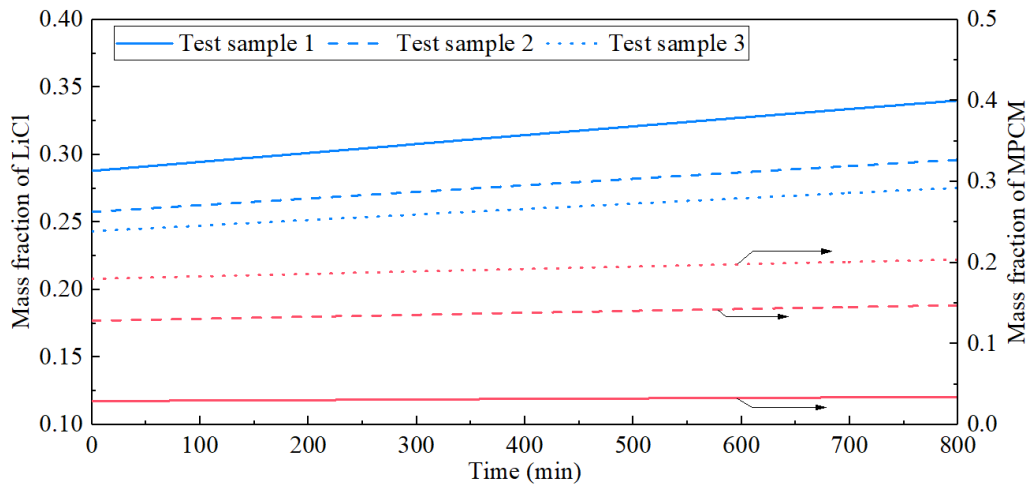


417

418 **Fig. 10.** Variation of vapour pressure of the PCM-LiCl desiccant solutions and pure LiCl

419

desiccant solutions as a function of time.



420

421 **Fig. 11.** Variation of mass fractions of the MPCM and LiCl in the PCM-LiCl desiccant

422

solutions as a function of time.

423

424 From the above results, it can be concluded that the dispersion of MPCMs into the liquid

425 desiccant solution can decrease the vapour pressure and increase the thermal capacity of the
426 mixture, which can improve the dehumidification efficiency of adiabatic dehumidifiers
427 although the thermal conductivity of the mixture was slightly decreased. In principle, a low
428 solution flow rate can be used in the liquid desiccant cooling system when using the PCM-LiCl
429 desiccant solution due to the increased thermal capacity and reduced vapour pressure. It is also
430 expected that the size of the dehumidifier using the new working solution can be decreased
431 without compromising the dehumidification performance, when compared to the dehumidifier
432 using pure desiccant solutions. In addition, the temperature of the inlet solution to the
433 dehumidifier should be optimised in order to maximise the benefits of using MPCMs in the
434 desiccant solution. It is also noteworthy that thermal regeneration might not be an optimal
435 method for liquid desiccant cooling systems when using PCM-LiCl desiccant solutions and
436 non-thermal regeneration methods may be required, which will be investigated in future studies.
437 However, the potential benefits of using PCM-LiCl desiccant solutions should be evaluated in
438 liquid desiccant cooling systems by considering the influence of the MPCM particles in both
439 dehumidifiers and regenerators. Adding the MPCM particles into liquid desiccant solutions
440 may increase the power consumptions of the circulation pump. Meanwhile, the mass fraction
441 of the MPCMs in the mixture should be optimised. The long-term performance and potential
442 segregation of MPCM particles in the new working solution should also be examined.

443 **4. Conclusions**

444 This study presented the development and characterisation of a new phase change enhanced
445 working solution for liquid desiccant cooling systems. The new solution was prepared through

446 dispersion of micro-encapsulated PCMs (i.e. Micronal DS 5038X and DS 5040X) into the base
447 fluid of the LiCl desiccant solution. The properties including density, h - T relationship, thermal
448 conductivity, particle size distribution, and vapour pressure of the new solution were
449 characterised either through direct measurement or theoretical analysis.

450 The results showed that the density of the phase change enhanced LiCl (PCM-LiCl)
451 desiccant solution decreased with the increase of the mass fraction of the MPCMs in the mixture
452 due to the lower density of the MPCMs used in comparison with the pure LiCl solutions. The
453 thermal capacity of the PCM-LiCl desiccant solutions was substantially increased in the melting
454 range of the MPCMs. The thermal conductivities of the PCM-LiCl desiccant solutions using
455 the MPCMs DS 5038X and DS 5040X were very close to each other when the mass fraction of
456 the MPCMs was less than 20% and the thermal conductivity decreased with the increase of the
457 mass fraction of the MPCMs. The vapour pressure of the new working solution decreased due
458 to the existence of the MPCM particle as compared to the LiCl solution without using the
459 MPCMs. The dehumidification performance of adiabatic dehumidifiers could be potentially
460 improved by using this new working solution due to its decreased vapour pressure and increased
461 thermal capacity, which will be investigated in our future studies.

462 **Acknowledgement**

463 The first author would like to thank German Academic Exchange Service (DAAD) for
464 providing financial support to visit Fraunhofer Institute for Solar Energy Systems, Freiburg,
465 Germany, to carry out this work.

466 **References**

- 467 [1] P.G. Dixon, M. Sinyor, A. Schaffer, A. Levitt, C.R Haney, K.N Ellis, S.C Sheridan,
468 Association of weekly suicide rates with temperature anomalies in two different climate types,
469 International Journal of Environmental Research and Public Health 11 (11) (2014) 11627-11644.
- 470 [2] H. Chappells, E. Shove, Debating the future of comfort: environmental sustainability,
471 energy consumption and the indoor environment, Building Research & Information 33 (1)
472 (2005) 32-40.
- 473 [3] R.C. Arora, Refrigeration and air conditioning, PHI Learning Pvt. Ltd., New Delhi, 2012.
- 474 [4] American Society of Heating, Refrigerating and Air-Conditioning Engineers (ASHRAE),
475 ASHRAE Handbook-Fundamentals, ASHRAE, Atlanta, 2009.
- 476 [5] L. Mei, Y.J. Dai, A technical review on use of liquid-desiccant dehumidification for air-
477 conditioning application, Renewable and Sustainable Energy Reviews 12 (3) (2008) 662-689.
- 478 [6] A.H. Abdel-Salam, C.J. Simonson, State-of-the-art in liquid desiccant air conditioning
479 equipment and systems, Renewable and Sustainable Energy Reviews 58 (2016) 1152-1183.
- 480 [7] Y. Guo, Z. Ma, A. Al-Jubainawi, P. Cooper, L.D. Nghiem, Using electro dialysis for
481 regeneration of aqueous lithium chloride solution in liquid desiccant air conditioning systems,
482 Energy and Buildings 116 (2016) 285-295.
- 483 [8] D.L. Shukla, K.V. Modi, A technical review on regeneration of liquid desiccant using solar
484 energy, Renewable and Sustainable Energy Reviews 78 (2017) 517-529.
- 485 [9] S.W. Ham, S.J. Lee, J.W. Jeong, Operating energy savings in a liquid desiccant and dew
486 point evaporative cooling-assisted 100% outdoor air system, Energy and Buildings 116 (2016)
487 535-552.

- 488 [10]T. Elmer, M. Worall, S. Wu, S. Riffat, An experimental study of a novel integrated desiccant
489 air conditioning system for building applications, *Energy and Buildings* 111 (2016) 434-445.
- 490 [11]Y. Chen, H. Yang, Y. Luo, Investigation on solar assisted liquid desiccant dehumidifier and
491 evaporative cooling system for fresh air treatment, *Energy* 143 (2018) 114-127.
- 492 [12]L-DCS Technology, <http://www.l-dcs.de/>, 2018 (accessed 15 December 2018)
- 493 [13]A. Hublitz, Efficient Energy Storage in Liquid Desiccant Cooling Systems, Ph.D Thesis,
494 Technische Universität München, München, Germany, 2008.
- 495 [14]M.M. Rafique, P. Gandhidasan, H.M. Bahaidarah, Liquid desiccant materials and
496 dehumidifiers—A review, *Renewable and Sustainable Energy Reviews* 56 (2016) 179-195.
- 497 [15]N. Fumo, D.Y. Goswami, Study of an aqueous lithium chloride desiccant system: air
498 dehumidification and desiccant regeneration, *Solar Energy* 72 (4) (2002) 351-361.
- 499 [16]A. Lowenstein, Review of liquid desiccant technology for HVAC applications, *HVAC&R*
500 *Research* 14 (6) (2008) 819-839.
- 501 [17]P. Bansal, S. Jain, C. Moon, Performance comparison of an adiabatic and an internally
502 cooled structured packed-bed dehumidifier, *Applied Thermal Engineering* 31 (1) (2011) 14-19.
- 503 [18]Z. Ma, W. Lin, M.I. Sohel, Nano-enhanced phase change materials for improved building
504 performance, *Renewable and Sustainable Energy Reviews* 58 (2016) 1256-1268.
- 505 [19]Z. Hu, A. Li, R. Gao, H. Yin, A comparison study on melting inside the rectangular and
506 curved unit with a vertical heating wall, *Journal of Thermal Analysis and Calorimetry* 122 (2)
507 (2015) 831-842.
- 508 [20]X. Niu, Q. Xu, Y. Zhang, Y. Zhang, Y. Yan, T. Liu, Fabrication and properties of micro-

509 nanoencapsulated phase change materials for internally-cooled liquid desiccant
510 dehumidification, *Nanomaterials* 7 (5) (2017) 96.

511 [21]A.A. Al-Abidi, S. Mat, K. Sopian, M.Y. Sulaiman, A. T. Mohammad, Experimental study
512 of PCM melting in triplex tube thermal energy storage for liquid desiccant air conditioning
513 system, *Energy and Buildings* 60 (2013) 270-279.

514 [22]J.M. Mahdi, E.C. Nsofor, Melting enhancement in triplex-tube latent heat energy storage
515 system using nanoparticles-metal foam combination, *Applied Energy* 191 (2017) 22-34.

516 [23]L. Wang, F. Xiao, X. Zhang, R. Kumar, An experimental study on the dehumidification
517 performance of a counter flow liquid desiccant dehumidifier, *International Journal of*
518 *Refrigeration* 70 (2016) 289-301.

519 [24]X.H. Liu, Y. Jiang, K.Y. Qu, Heat and mass transfer model of cross flow liquid desiccant
520 air dehumidifier/regenerator, *Energy Conversion and Management* 48 (2) (2007) 546-554.

521 [25]I.P. Koronaki, R.I. Christodoulaki, V.D. Papaefthimiou, E.D. Rogdakis, Thermodynamic
522 analysis of a counter flow adiabatic dehumidifier with different liquid desiccant materials,
523 *Applied Thermal Engineering* 50 (1) (2013) 361-373.

524 [26]M.R. Islam, S.W.L. Alan, K.J. Chua, Studying the heat and mass transfer process of liquid
525 desiccant for dehumidification and cooling, *Applied Energy* 221 (2018) 334-347.

526 [27]B. Chen, X. Wang, R. Zeng, Y. Zhang, X. Wang, J. Niu, Y. Li, H. Di, An experimental study
527 of convective heat transfer with microencapsulated phase change material suspension: laminar
528 flow in a circular tube under constant heat flux, *Experimental Thermal and Fluid Science* 32 (8)
529 (2008) 1638-1646.

530 [28]M.R. Conde, Properties of aqueous solutions of lithium and calcium chlorides:
531 formulations for use in air conditioning equipment design, International Journal of Thermal
532 Sciences 43 (4) (2004) 367-382.

533 [29]S.K. Chaudhari, K.R Patil, Thermodynamic properties of aqueous solutions of lithium
534 chloride, Physics and Chemistry of Liquids 40 (3) (2002) 317-325.

535 [30]M. Goel, S.K. Roy, S. Sengupta, Laminar forced convection heat transfer in
536 microcapsulated phase change material suspensions, International Journal of Heat and Mass
537 Transfer 37 (4) (1994) 593-604.

538 [31]E.C. Guyer, Handbook of applied thermal design, CRC Press, Philadelphia, 1999.

539 [32]D.M. Price, Vapor pressure determination by thermogravimetry, Thermochemica Acta (367)
540 (2001) 253-262.

541 [33]S. Giani, Determination of vapor pressure and the enthalpy of vaporization by TGA,
542 METTLER TOLEDO Thermal Analysis UserCom 38 (2014) 15-18.

543 [34]Micronal Product Manual, <https://www.microteklabs.com/micronal>, 2018 (accessed 10
544 December 2018).

545 [35]D.L. Black, M.Q. McQuay, M.P. Bonin, Laser-based techniques for particle-size
546 measurement: a review of sizing methods and their industrial applications, Progress in Energy
547 and Combustion Science 22 (3) (1996) 267-306.

548 [36]G.B. de Boer, C. de Weerd, D. Thoenes, H.W. Goossens, Laser diffraction spectrometry:
549 Fraunhofer diffraction versus Mie scattering, Particle & Particle Systems Characterization 4 (1-
550 4) (1987) 14-19.

551 [37] J. Giro-Paloma, G. Oncins, C. Barreneche, M. Martínez, A.I. Fernández, L.F. Cabeza,
552 Physico-chemical and mechanical properties of microencapsulated phase change material,
553 Applied Energy 109 (2013) 441-448.

554 [38]P. Masset, Thermogravimetric study of the dehydration reaction of $\text{LiCl}\cdot\text{H}_2\text{O}$, Journal of
555 Thermal Analysis and Calorimetry 96 (2) (2009) 439-441.

556 [39]M.C. Costache, D. Wang, M.J. Heidecker, E. Manias, C.A. Wilkie, The thermal degradation
557 of poly (methyl methacrylate) nanocomposites with montmorillonite, layered double
558 hydroxides and carbon nanotubes, Polymers for Advanced Technologies 17 (4) (2006) 272-280.

559 [40]J. Giro-Paloma, C. Barreneche, M. Delgado, M. Martínez, A.I. Fernández, L.F. Cabeza,
560 Physicochemical and thermal study of a MPCM of PMMA shell and paraffin wax as a core,
561 Energy Procedia 54 (2014) 347-354.