

Research Article

Adaptive Fuzzy Integral Terminal Sliding Mode Control of a Nonholonomic Wheeled Mobile Robot

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In this paper, the trajectory tracking problem is investigated for a nonholonomic wheeled mobile robot with parameter uncertainties and external disturbances. In this strategy, combining the kinematic model with the dynamic model, the actuator voltage is employed as the control input, and the uncertainties are approximated by a fuzzy logic system. An auxiliary velocity controller is integrated with an adaptive fuzzy integral terminal sliding mode controller, and a robust controller is employed to compensate for the lumped errors. It is proved that all the signals in the closed system are bounded and the auxiliary velocity tracking errors can converge to a small neighborhood of the origin in finite time. As a result, the tracking position errors converge asymptotically to zeros with faster response than other existing controllers. Simulation results demonstrate the effectiveness of the proposed strategy.

1. Introduction

A wheeled mobile robot (WMR) is an uncertain nonlinear MIMO dynamic system. When the WMR constrains the wheel's "pure rolling without slipping," it is also a typical kind of nonholonomic systems characterized by kinematic constraints. Such constraints are not integrable and can not be eliminated from the model equations. Given so many characteristics that are hard to handle, there has been tremendous research on the nonholonomic WMR (NWMR) in the past few decades.

The trajectory tracking problem is one of the most popular problems on the WMR. With an assumption of "perfect velocity tracking," the initial kinematic controller for the NWMR was designed in [1, 2]. However, such an assumption is difficult to hold in practice for the dynamic model of the NWMR is neglected. Considering the kinematic model and the dynamic model of the NWMR together, based on backstepping technique, Fierro and Lewis [3] presented a dynamical extension that combines a kinematic controller with a torque controller. In this method, it is assumed that the dynamic structure of the NWMR and the parameters

are completely known. However, in practical WMRs, there exist parameter uncertainties and external disturbances. In addition, wheel skidding and slipping may happen. To overcome these difficulties, the torus shaped rear wheels were used for three WMRs in [4, 5]. The modeling and analysis were investigated to design the controller for the WMR in [6]. Meanwhile, a variety of nonlinear control techniques have been used by many researchers, such as adaptive control [7–11], robust adaptive control [12–14], adaptive fuzzy logic control [15–18], adaptive neural network control [19, 20], and sliding mode control [21–23], and several kinds of the aforementioned methodologies are integrated to solve this problem [24, 25].

One idea of some proposed literatures related to the trajectory tracking problem of the NWMR is that an auxiliary velocity controller is designed for the kinematic model of the NWMR to make the tracking position errors converge asymptotically to zeros, and a dynamic controller is designed for the dynamic model of the NWMR to make the auxiliary velocity tracking errors as small as possible. Meanwhile, a robust controller is employed to compensate the total uncertainties. For instance, by virtue of the universal

approximation property of the fuzzy logic system (FLS) [26–31], a control structure combining a kinematic controller with a dynamic controller plus a fuzzy compensator was proposed in [15]. A complete control law based on a kinematic controller and an adaptive fuzzy sliding mode controller was developed for a NWMR in the presence of dynamic uncertainties as well in [25]. These dynamic controllers share a common idea of choosing the wheel torque as the control input. However, as stated in [17], the wheel is driven by the actuator in reality. Hence, the resulting electrically driven mobile robot (the robot kinematics, robot dynamics, and wheel actuator dynamics) is represented as a third-order system. So most of existing torque controllers designed with respect to the second-order, that is, the wheel actuator dynamics, have been neglected and might degrade the performance of the tracking control. Therefore, it is more reasonable to use the actuator voltage as the control input. For realizing the trajectory tracking of the NWMR with high performance, the wheel actuator dynamics are combining with the dynamics of the NWMR and the actuator voltage is employed as the control input in [8, 16–18]. All these dynamic controllers can guarantee that the auxiliary velocity tracking errors converge to an adjustable neighborhood of the origin as time goes to infinity. However, the finite time convergence of the auxiliary velocity tracking errors can not be guaranteed.

The terminal sliding mode control (TSMC), which was first proposed in [32, 33], is an effective scheme to guarantee the finite time convergence of the auxiliary velocity tracking errors. However, the initial TSMC may cause the singularity problem around the equilibrium [34], which would result in an unbounded control signal. In order to avoid this problem, a nonsingular terminal sliding mode control (NTSMC) was developed in [35–38]. The continuous nonsingular terminal sliding mode [36] has been extended into a class of MIMO nonlinear systems [39]. Furthermore, using integral operation, an integral terminal sliding mode control (ITSMC) was presented in [40, 41] for a class of first-order systems. Apart from finite time convergence and nonsingularity, in the ITSMC design, the system can also start on the integral terminal sliding mode surface from the initial time instant. Therefore, the reaching time to the sliding mode surface is eliminated.

Based on the previous results, this paper addresses the trajectory tracking problem for the NWMR with parameter uncertainties and external disturbances. Combining the kinematic model with the dynamic model, a control strategy is proposed which integrates an auxiliary velocity controller with an adaptive fuzzy integral terminal sliding mode controller. In this control strategy, using the universal approximation property of the FLS, the uncertainties are approximated by a fuzzy logic system and a robust controller is employed to compensate for the lumped errors. Meanwhile, instead of the wheel torque, the actuator voltage is employed as the control input. The main originality of the proposed control strategy is that the adaptive fuzzy integral terminal sliding mode controller can guarantee the finite time convergence of the auxiliary velocity tracking errors. It is proved that all the signals in the closed system are bounded and the auxiliary velocity tracking errors converge to a small

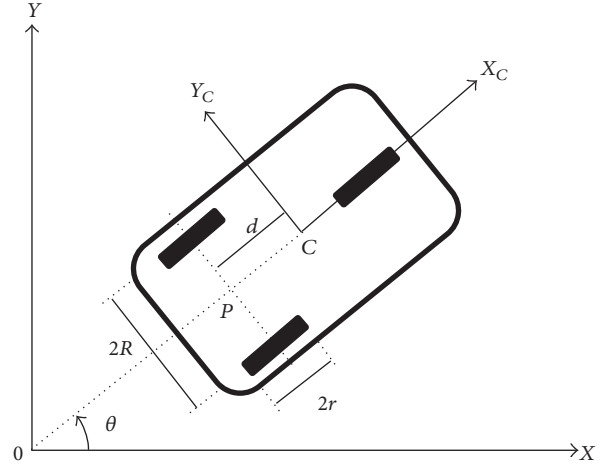


FIGURE 1: A wheeled mobile robot.

neighborhood of the origin in finite time. Therefore, the tracking position errors converge asymptotically to zeros with faster response than other existing controllers. Simulation results demonstrate the effectiveness of the proposed strategy.

The remainder of this paper is organized as follows. Section 2 reviews some basics of the model of the NWMR, the ITSMC, and the FLS. By use of the ITSMC and the FLS, a control strategy is proposed which integrates an auxiliary velocity controller with an adaptive fuzzy integral terminal sliding mode controller in Section 3. Section 4 gives simulation results to illustrate our results. Conclusions are given in Section 5.

2. Preliminaries

In this section, we will review some basics of the model of the NWMR, the ITSMC, and the FLS briefly.

2.1. Model of the Nonholonomic Wheeled Mobile Robot. We consider a typical example of the WMR, which is called Type (2,0) WMR in [6]. Such a WMR is composed of two driving wheels and one passive wheel. The two driving wheels are controlled independently by two actuators to achieve the motion and orientation, and the passive wheel prevents the robot from tipping over as it moves on a plane. Figure 1 describes the posture of the WMR in Cartesian coordinates. Both driving wheels with the same radius r are mounted on the same axis and separated by $2R$. The center of mass of the WMR is located at C , and P is located in the midpoint of the two driving wheels. The distance between P and C is d . When the electrical part of the actuator is taken into account, the kinematic equation and the dynamic equation of the NWMR can be written as follows from [16, 19]:

$$\dot{q} = s(q) \vartheta, \quad (1)$$

$$\begin{aligned} \overline{M}(q) \dot{\vartheta} + \overline{V}(q, \dot{q}) \vartheta + \overline{F}(\vartheta) + \overline{\tau}_d \\ = \frac{NK_T}{R_a} \overline{B}u - \frac{N^2 K_T K_b}{R_a} \overline{B}X \vartheta, \end{aligned} \quad (2)$$

where

$$\begin{aligned} S(q) &= \begin{bmatrix} \cos \theta & -d \sin \theta \\ \sin \theta & d \cos \theta \\ 0 & 1 \end{bmatrix}, \\ \bar{M}(q) &= \begin{bmatrix} m & 0 \\ 0 & I - md^2 \end{bmatrix}, \\ \bar{V}(q, \dot{q}) &= \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}, \\ \bar{B} &= \frac{1}{r} \begin{bmatrix} 1 & 1 \\ R & -R \end{bmatrix}, \\ X &= \bar{B}^T. \end{aligned} \quad (3)$$

$q = [x, y, \theta]^T$, $C(x, y)$ is the coordinate of C in the global coordinate frame XOY , and θ is the orientation of the local coordinate frame $X_C Y_C$ attached on the WMR platform measured from X axis and is also called the heading angle of the WMR. $\vartheta = [\nu, \omega]^T$, where ν and ω are the linear velocity of the point P along the robot axis and angle velocity, respectively. $\bar{M}(q)$ is the inertia matrix, $\bar{V}(q, \dot{q})$ is the centripetal and Coriolis matrix, $\bar{F}(\vartheta) \in R^{2 \times 1}$ is the surface friction, and $\bar{\tau}_d \in R^{2 \times 1}$ denotes bounded unknown disturbances including unstructured unmodeled dynamics. N is the gear ration, K_T is the motor torque constant, K_b is the counter electromotive force coefficient, and R_a is the electric resistance. $u = [u_1, u_2]^T$ is the actuator voltage input vector.

Several properties of the NWMR are given as follows [19].

Property 1. The matrix $\bar{M}(q)$ is symmetric and positive definite.

Property 2. The matrix $\bar{M}(q)$ is bounded; that is, there exist positive constants m_1 and m_2 satisfying $m_1 \|x\|^2 \leq x^T \bar{M}(q)x \leq m_2 \|x\|^2$, for all $x \in R^2$.

Property 3. The matrix $\bar{M}(q) - 2\bar{V}(q, \dot{q})$ is skew symmetric resulting in the following characteristic: $x^T(\bar{M}(q) - 2\bar{V}(q, \dot{q}))x = 0$ for all $x \in R^2$.

In view of the dynamic model of the NWMR, (2) is a first-order system; the ITSM can be utilized so that the finite time convergence of the auxiliary velocity tracking errors of the NWMR is obtained.

2.2. Integral Terminal Sliding Mode. Now, a new form of the integral terminal sliding mode is defined as

$$s = x(t) - x(0) + \beta \int_0^t |x(\tau)|^\gamma \text{sign}(x(\tau)) d\tau, \quad (4)$$

where $x \in R$ is the system state variable, $\beta > 0$, and $0 < \gamma < 1$, which generalizes the integral terminal sliding mode [41]

$$s = x(t) + \beta \int_0^t x^{q/p}(\tau) d\tau, \quad (5)$$

where $\beta > 0$ and p and q are odd integers satisfying $p > q > 0$.

Remark 1. It is worthwhile to notice that the range of the power γ is larger than that of the power q/p . Meanwhile, by means of the basic theorem of differential and integral calculus [42], the integral terminal sliding mode (4) is continuous and differentiable although the absolute and signum operators are involved. Besides these properties, from (4), it is obvious that $s(0) = 0$ without the prior knowledge of the parameter β . This implies that the system starts on the integral terminal sliding mode surface (4) from the initial time instant much easily.

Furthermore, on the sliding surface, $s = 0$, which results in

$$\dot{x}(t) + \beta |x(t)|^\gamma \text{sign}(x(t)) = 0. \quad (6)$$

The finite time t_s that is taken from $x(0) \neq 0$ to $x(t_s) = 0$ is given by

$$t_s = \frac{1}{\beta(1-\gamma)} |x(0)|^{1-\gamma}. \quad (7)$$

As we stated previously, there exist parameter uncertainties and unknown disturbances in practical WMR. Taking these factors into account, an unknown nonlinear function is contained in the model of the NWMR. We will use the FLS to approximate this function.

2.3. Fuzzy Logic Systems. In this section, the FLS is discussed briefly. The basic configuration of an FLS consists of four components: fuzzifier, fuzzy rule base, fuzzy inference engine, and defuzzifier. The fuzzy rule base is a collection of IF-THEN rules and the l th fuzzy rule is written as

$$R^l: \text{IF } x_1 \text{ is } F_1^l \text{ and } \dots \text{and } x_n \text{ is } F_n^l, \text{ THEN } y \text{ is } G^l,$$

where F_i^l and G^l are fuzzy sets, associating with fuzzy membership functions $\mu_{F_i^l}(x_i)$ and $\mu_{G^l}(y)$, respectively, $i = 1, \dots, n$, $l = 1, \dots, m$, m is the number of rules.

Based on these fuzzy IF-THEN rules, the FLS performs a mapping from an input vector $x = [x_1, \dots, x_n]^T \in R^n$ to an output variable $y \in R$. If we use the strategy of singleton fuzzifier, product inference, and center-average defuzzifier, the output of the FLS can be defined as follows:

$$y(x) = \frac{\sum_{l=1}^m y^l \left(\prod_{i=1}^n \mu_{F_i^l}(x_i) \right)}{\sum_{l=1}^m \prod_{i=1}^n \mu_{F_i^l}(x_i)}, \quad (8)$$

where y^l is the point in G^l at which $\mu_{G^l}(y)$ obtains its maximum value 1.

For simplicity, $y(x)$ can be written in the following compact form:

$$y(x) = \hat{\theta}^T \xi(x) := \hat{f}(x | \hat{\theta}), \quad (9)$$

where $\hat{\theta} = [y^1, \dots, y^m]^T$ is called the unknown parameter vector which is to be updated and $\xi(x) = [\xi^1(x), \dots, \xi^m(x)]^T$ is called the fuzzy basis function vector, $\xi^l(x) = \prod_{i=1}^n \mu_{F_i^l}(x_i) / \sum_{l=1}^m \prod_{i=1}^n \mu_{F_i^l}(x_i)$, $l = 1, 2, \dots, m$.

Lemma 2 (see [31]). *Let $f(x)$ be a continuous function defined on a compact set Ω . Then, for any constant $\varepsilon > 0$, there exists a fuzzy system (9) such that $\sup_{x \in \Omega} |f(x) - \hat{f}(x | \hat{\theta})| \leq \varepsilon$.*

3. Controller Design

It is easy to see that the posture of the NWMR $q(t) = [x(t), y(t), \theta(t)]^T$ satisfies the following equations from (1):

$$\begin{aligned}\dot{x} &= v \cos \theta - \omega d \sin \theta, \\ \dot{y} &= v \sin \theta + \omega d \cos \theta, \\ \dot{\theta} &= \omega.\end{aligned}\quad (10)$$

It is assumed that the reference trajectory $q_r(t) = [x_r(t), y_r(t), \theta_r(t)]^T$ is generated by a reference NWMR with the kinematic equation as (10):

$$\begin{aligned}\dot{x}_r &= v_r \cos \theta_r - \omega_r d \sin \theta_r, \\ \dot{y}_r &= v_r \sin \theta_r + \omega_r d \cos \theta_r, \\ \dot{\theta}_r &= \omega_r.\end{aligned}\quad (11)$$

The objective of the trajectory tracking control is to design a strategy such that $q(t)$ converges asymptotically to $q_r(t)$, while all signals in the derived closed-loop system remain bounded. In this study, an auxiliary velocity controller ϑ_c is designed for the kinematic model (1) to meet the control objective. Then, the actuator voltage control input u is designed for the dynamic model (2) such that ϑ converges to ϑ_c which is designed at the first step in finite time.

Remark 3. As pointed out in [18], the classical auxiliary velocity controller [1] adopted in [3, 16, 18, 19, 25] can only guarantee that $q(t)$ converges asymptotically to $q_r(t)$ when d equals zero. However, d does not equal zero in general. Therefore, the reference point of the practical NWMR is not in accordance with the desired point of the reference NWMR, which results in incomplete tracking of the posture. In this paper, we modify the kinematic model of the reference NWMR as (11) and adopt another auxiliary velocity controller [43].

3.1. An Auxiliary Velocity Controller Design. We define the tracking position errors as the difference between the center of mass C of the NWMR and the desired point of the reference NWMR as follows [16, 19]:

$$e_p = \begin{bmatrix} e_1 \\ e_2 \\ e_3 \end{bmatrix} = \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_r - x \\ y_r - y \\ \theta_r - \theta \end{bmatrix}. \quad (12)$$

The first derivative of the error yields

$$\begin{aligned}\begin{bmatrix} \dot{e}_1 \\ \dot{e}_2 \\ \dot{e}_3 \end{bmatrix} &= \begin{bmatrix} -1 & e_2 \\ 0 & -e_1 - d \\ 0 & -1 \end{bmatrix} \begin{bmatrix} v \\ \omega \end{bmatrix} \\ &+ \begin{bmatrix} v_r \cos e_3 - \omega_r d \sin e_3 \\ v_r \sin e_3 + \omega_r d \cos e_3 \\ \omega_r \end{bmatrix}.\end{aligned}\quad (13)$$

Therefore, the objective of this study becomes the design of an auxiliary velocity controller to make the tracking position errors asymptotically converge to zeros. In this study, according to [43], the auxiliary velocity controller is designed as

$$\vartheta_c = \begin{bmatrix} v_c \\ \omega_c \end{bmatrix} = \begin{bmatrix} v_r \cos e_3 + k_1 (e_1 + d - d \cos e_3) \\ \omega_r + k_3 v_r (e_2 - d \sin e_3) + k_2 \sin e_3 \end{bmatrix}, \quad (14)$$

where $k_1, k_2, k_3 > 0$ are design parameters.

Substituting (14) into (13), the closed-loop kinematic equation can be written as

$$\begin{aligned}\begin{bmatrix} \dot{e}_1 \\ \dot{e}_2 \\ \dot{e}_3 \end{bmatrix} &= \begin{bmatrix} -1 & e_2 \\ 0 & -e_1 - d \\ 0 & -1 \end{bmatrix} \begin{bmatrix} v_r \cos e_3 + k_1 (e_1 + d - d \cos e_3) \\ \omega_r + k_3 v_r (e_2 - d \sin e_3) + k_2 \sin e_3 \end{bmatrix} \\ &+ \begin{bmatrix} v_r \cos e_3 - \omega_r d \sin e_3 \\ v_r \sin e_3 + \omega_r d \cos e_3 \\ \omega_r \end{bmatrix}.\end{aligned}\quad (15)$$

Assumption 4 (see [8]). The reference velocities $\vartheta_r = [v_r, \omega_r]^T$ and \dot{v}_r are bounded.

Lemma 5. *For the kinematic model (1) of the NWMR satisfying Assumption 4, the auxiliary velocity controller (14) will ensure that the tracking position errors converge asymptotically to zeros.*

Proof. Consider the following Lyapunov function candidate

$$\begin{aligned}V_1 &= \frac{1}{2} (e_1 + d - d \cos e_3)^2 + \frac{1}{2} (e_2 - d \sin e_3)^2 \\ &+ \frac{1 - \cos e_3}{k_3}.\end{aligned}\quad (16)$$

Differentiating V_1 with respect to time, we have

$$\begin{aligned}\dot{V}_1 &= (e_1 + d - d \cos e_3) (\dot{e}_1 + d \dot{e}_3 \sin e_3) \\ &+ (e_2 - d \sin e_3) (\dot{e}_2 - d \dot{e}_3 \cos e_3) + \frac{1}{k_3} \dot{e}_3 \sin e_3.\end{aligned}\quad (17)$$

Replacing (15) into (17) and after some manipulations, one obtains

$$\dot{V}_1 = -k_1 (e_1 + d - d \cos e_3)^2 - \frac{k_2}{k_3} \sin^2 e_3 \leq 0. \quad (18)$$

Therefore, the tracking position error $e_p = [e_1, e_2, e_3]^T$ is bounded. With Assumption 4, ϑ_c and $\dot{e}_p = [\dot{e}_1, \dot{e}_2, \dot{e}_3]^T$ are bounded. So \dot{V}_1 is bounded and \dot{V}_1 is uniformly continuous accordingly. By Barbalat's lemma [44], $\dot{V}_1 \rightarrow 0$ as $t \rightarrow \infty$, which implies that $e_1 \rightarrow 0$ and $e_3 \rightarrow 0$ as $t \rightarrow \infty$.

From (15), one obtains

$$\dot{e}_3 = -k_3 v_r (e_2 - d \sin e_3) - k_2 \sin e_3. \quad (19)$$

Using Barbalat's lemma again, $\dot{e}_3 \rightarrow 0$ as $t \rightarrow \infty$, which implies that $e_2 \rightarrow 0$ as $t \rightarrow \infty$.

Hence, $e_p \rightarrow 0$ as $t \rightarrow \infty$; that is, the tracking position errors converge asymptotically to zeros. \square

Now, it remains to design the actuator voltage control input so that the desired velocities ϑ_c can be obtained in finite time.

3.2. Adaptive Integral Terminal Sliding Mode Controller Design. In this study, the auxiliary velocity tracking error is defined as

$$e_\vartheta = [e_{\vartheta 1}, e_{\vartheta 2}]^T = \vartheta_c - \vartheta. \quad (20)$$

Consequently, the dynamic equation (2) of the NWMR can be rewritten as

$$\begin{aligned} \overline{M}(q) \dot{e}_\vartheta &= -\overline{V}(q, \dot{q}) e_\vartheta + \frac{N^2 K_T K_b}{R_a} \overline{B} X \vartheta + \overline{M}(q) \dot{\vartheta}_c \\ &+ \overline{V}(q, \dot{q}) \vartheta_c + \overline{F}(\vartheta) + \overline{\tau}_d - \frac{N K_T}{R_a} \overline{B} u. \end{aligned} \quad (21)$$

A continuous nonsingular integral terminal sliding mode is defined as in the form (4):

$$\begin{aligned} S &= \begin{bmatrix} s_1 \\ s_2 \end{bmatrix} \\ &= \begin{bmatrix} e_{\vartheta 1} - e_{\vartheta 1}(0) + \beta_1 \int_0^t |e_{\vartheta 1}|^{\gamma_1} \text{sign}(e_{\vartheta 1}) d\tau \\ e_{\vartheta 2} - e_{\vartheta 2}(0) + \beta_2 \int_0^t |e_{\vartheta 2}|^{\gamma_2} \text{sign}(e_{\vartheta 2}) d\tau \end{bmatrix}, \end{aligned} \quad (22)$$

where $\beta_i > 0$, $0 < \gamma_i < 1$, $i = 1, 2$.

Denote

$$\Lambda = \text{diag}(\beta_1, \beta_2), \quad (23)$$

$$\text{sig}(e_\vartheta)^\gamma = [|e_{\vartheta 1}|^{\gamma_1} \text{sign}(e_{\vartheta 1}), |e_{\vartheta 2}|^{\gamma_2} \text{sign}(e_{\vartheta 2})]^T.$$

S can be rewritten as follows:

$$S = e_\vartheta - e_\vartheta(0) + \Lambda \int_0^t \text{sig}(e_\vartheta)^\gamma d\tau. \quad (24)$$

Utilizing S and its derivative with respect to time, (21) can be arranged as follows:

$$\begin{aligned} \overline{M}(q) \dot{S} &= -\overline{V}(q, \dot{q}) S + \frac{N^2 K_T K_b}{R_a} \overline{B} X \vartheta + f(x) \\ &- \frac{N K_T}{R_a} \overline{B} u, \end{aligned} \quad (25)$$

where

$$\begin{aligned} f(x) &= [f_1(x), f_2(x)]^T \\ &= \overline{M}(q) \dot{\vartheta}_c + \overline{M}(q) \Lambda \text{sig}(e_\vartheta)^\gamma + \overline{V}(q, \dot{q}) \vartheta_c \\ &- \overline{V}(q, \dot{q}) e_\vartheta(0) + \overline{V}(q, \dot{q}) \Lambda \int_0^t \text{sig}(e_\vartheta)^\gamma d\tau \\ &+ \overline{F}(\vartheta) + \overline{\tau}_d \end{aligned} \quad (26)$$

and $x = [\vartheta_c^T, \dot{\vartheta}_c^T, \vartheta^T]^T$.

If $f(x)$ is known, let the actuator voltage control input

$$\begin{aligned} u &= \frac{R_a}{N K_T} \overline{B}^{-1} \left(\frac{N^2 K_T K_b}{R_a} \overline{B} X \vartheta + f(x) + K_1 S \right. \\ &\left. + K_2 \text{sig}(S)^\rho \right), \end{aligned} \quad (27)$$

where $K_1 = \text{diag}(k_{11}, k_{12})$, $K_2 = \text{diag}(k_{21}, k_{22})$, $k_{1i} > 0$, $k_{2i} > 0$, $i = 1, 2$, and $0 < \rho < 1$.

Substituting (27) into (25), the closed-loop dynamic equation can be written as

$$\overline{M}(q) \dot{S} = -\overline{V}(q, \dot{q}) S - K_1 S - K_2 \text{sig}(S)^\rho. \quad (28)$$

Multiplying S^T by (28) yields

$$S^T \overline{M}(q) \dot{S} = -S^T \overline{V}(q, \dot{q}) S - S^T K_1 S - S^T K_2 \text{sig}(S)^\rho. \quad (29)$$

Theorem 6. For the dynamic model (2) of the NWMR with a known function $f(x)$ (26), if the integral terminal sliding mode is chosen as (22) and the actuator voltage control input is designed as (27), then the integral terminal sliding mode S and the auxiliary velocity tracking error e_ϑ will converge to zeros in finite time.

To prove Theorem 6, we introduce two lemmas.

Lemma 7 (see [36]). Suppose a_1, a_2, \dots, a_n and $0 < p < 2$ are all positive numbers, then the following inequality holds:

$$(a_1^2 + a_2^2 + \dots + a_n^2)^p \leq (a_1^p + a_2^p + \dots + a_n^p)^2. \quad (30)$$

Lemma 8 (see [36]). An extended Lyapunov description of finite time stability can be given with the form of fast terminal sliding mode as

$$\dot{V}(x) + \alpha V(x) + \beta V^\gamma(x) \leq 0, \quad \alpha, \beta > 0, \quad 0 < \gamma < 1, \quad (31)$$

and the settling time can be given by

$$t_r \leq \frac{1}{\alpha(1-\gamma)} \ln \frac{\alpha V^{1-\gamma}(x(0)) + \beta}{\beta}. \quad (32)$$

Proof of Theorem 6. Consider the following Lyapunov function candidate:

$$V_2 = \frac{1}{2} S^T \bar{M}(q) S. \quad (33)$$

Differentiating V_2 with respect to time and using (29) yields

$$\begin{aligned} \dot{V}_2 &= S^T \bar{M}(q) \dot{S} + \frac{1}{2} S^T \dot{\bar{M}}(q) S \\ &= \frac{1}{2} S^T \left(\dot{\bar{M}}(q) - 2\bar{V}(q, \dot{q}) \right) S - S^T K_1 S \\ &\quad - S^T K_2 \text{sig}(S)^\rho. \end{aligned} \quad (34)$$

From Property 3, which makes the first term zero, \dot{V}_2 becomes

$$\dot{V}_2 = -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho. \quad (35)$$

Denote $\lambda_i = \min\{k_{i1}, k_{i2}\}$, $i = 1, 2$; the following inequality holds:

$$\dot{V}_2 \leq -\lambda_1 \sum_{i=1}^2 s_i^2 - \lambda_2 \sum_{i=1}^2 |s_i|^{\rho+1}. \quad (36)$$

Applying Lemma 7 into (36), one obtains

$$\begin{aligned} \dot{V}_2 &\leq -\lambda_1 \frac{2}{m_2} \sum_{i=1}^2 \frac{m_2}{2} s_i^2 \\ &\quad - \lambda_2 \left(\frac{2}{m_2} \right)^{(\rho+1)/2} \left[\sum_{i=1}^2 \frac{m_2 s_i^2}{2} \right]^{(\rho+1)/2}. \end{aligned} \quad (37)$$

Utilizing Property 2, we have

$$\dot{V}_2 \leq -\lambda_1 \frac{2}{m_2} V_2 - \lambda_2 \left(\frac{2}{m_2} \right)^{(\rho+1)/2} V_2^{(\rho+1)/2}, \quad (38)$$

or

$$\dot{V}_2 + \lambda_1 \frac{2}{m_2} V_2 + \lambda_2 \left(\frac{2}{m_2} \right)^{(\rho+1)/2} V_2^{(\rho+1)/2} \leq 0. \quad (39)$$

From Lemma 8, it follows that S will converge to zero in finite time

$$\begin{aligned} t_r &\leq \frac{1}{\lambda_1 (2/m_2) (1 - (\rho+1)/2)} \\ &\quad \cdot \ln \frac{\lambda_1 (2/m_2) V_2^{1-(\rho+1)/2}(0) + \lambda_2 (2/m_2)^{(\rho+1)/2}}{\lambda_2 (2/m_2)^{(\rho+1)/2}}. \end{aligned} \quad (40)$$

Moreover, on the sliding mode surface, according to (7),

$$t_{si} = \frac{1}{\beta_i (1 - \gamma_i)} |e_{\theta_i}(0)|^{1-\gamma_i}, \quad i = 1, 2. \quad (41)$$

Therefore, the auxiliary velocity tracking error e_θ will converge to zero in finite time $t = t_r + \max\{t_{s1}, t_{s2}\}$. \square

Due to the fact that $f(x)$ contains all the mobile robot parameters (such as mass, moment of inertia, and friction coefficients) and external disturbances, in the following, we assume that $f_i(x)$, $i = 1, 2$, can be approximated by the following FLS:

$$\hat{f}_i(x | \hat{\theta}_{fi}) = \hat{\theta}_{fi}^T \xi_{fi}(x), \quad (42)$$

where $\xi_{fi}(x)$ is the fuzzy basis function vector and $\hat{\theta}_{fi}$ is the parameter vector of each fuzzy system designed later.

Define the optimal approximation parameters $\hat{\theta}_{fi}^*$ as follows:

$$\hat{\theta}_{fi}^* = \underset{\hat{\theta}_{fi} \in \Omega_{fi}}{\text{argmin}} \left[\sup_{x \in U} |f_i(x) - \hat{f}_i(x | \hat{\theta}_{fi})| \right], \quad (43)$$

where Ω_{fi} is the compact set of allowable controller parameters. Moreover, the parameter error and the minimum approximation error are defined as $\bar{\theta}_{fi} = \hat{\theta}_{fi}^* - \hat{\theta}_{fi}$ and $\omega_{fi}(x) = f_i(x) - \hat{f}_i(x | \hat{\theta}_{fi}^*)$, respectively.

Assumption 9. For $i = 1, 2$, ω_{fi} is bounded. That is, there exists an unknown constant $\bar{\omega}_{fi}$ such that $|\omega_{fi}(x)| < \bar{\omega}_{fi}$.

Denote

$$\begin{aligned} \hat{f}(x | \hat{\theta}_f) &= [\hat{f}_1(x | \hat{\theta}_{f1}), \hat{f}_2(x | \hat{\theta}_{f2})]^T, \\ \hat{f}(x | \theta_f^*) &= [\hat{f}_1(x | \theta_{f1}^*), \hat{f}_2(x | \theta_{f2}^*)]^T. \end{aligned} \quad (44)$$

By using the fuzzy approximation $\hat{f}(x | \hat{\theta}_f)$ instead of $f(x)$, the following control law from (27) is obtained:

$$\begin{aligned} u &= \frac{R_a}{NK_T} \bar{B}^{-1} \left(\frac{N^2 K_T K_b}{R_a} \bar{B} X \dot{\theta} + \hat{f}(x | \hat{\theta}_f) + K_1 s \right. \\ &\quad \left. + K_2 \text{sig}(s)^\rho + u_r \right), \end{aligned} \quad (45)$$

where $u_r = [u_{r1}, u_{r2}]^T$ is a robust controller, which is designed as

$$u_r = \begin{cases} \frac{(\hat{\varepsilon}_f + \sigma)s}{\|s\|}, & s \neq 0, \\ 0, & s = 0, \end{cases} \quad (46)$$

$\hat{\varepsilon}_f$ is the estimate of $\sum_{i=1}^2 \bar{\omega}_{fi}$, $\tilde{\varepsilon}_f = \hat{\varepsilon}_f - \sum_{i=1}^2 \bar{\omega}_{fi}$, and σ is a positive constant.

Remark 10. It is noticed that the robust controller u_r (46) is similar to that in [37]. However, in [37], σ is required no less than the estimation error of the unknown function. Whereas, in practice, it is difficult to determine such an estimation error. In this paper, σ is relaxed to be an arbitrary positive number.

Substituting (45) into (25), the closed-loop dynamic equation can be rewritten as

$$\begin{aligned} \bar{M}(q) \dot{S} &= -\bar{V}(q, \dot{q}) S - \hat{f}(x | \hat{\theta}) - K_1 S - K_2 \text{sig}(S)^\rho \\ &\quad - u_r + f(x). \end{aligned} \quad (47)$$

Multiplying S^T to (47) and after some manipulations, we can get

$$\begin{aligned} S^T \overline{M}(q) \dot{S} = & -S^T \overline{V}(q, \dot{q}) S - S^T K_1 S - S^T K_2 \text{sig}(S)^\rho \\ & - S^T u_r + \sum_{i=1}^2 s_i \omega_{fi}(x) + \sum_{i=1}^2 s_i \tilde{\theta}_{fi}^T \xi_{fi}(x). \end{aligned} \quad (48)$$

We use the following adaptation laws to adjust the unknown parameters $\hat{\theta}_{fi}$ and $\hat{\varepsilon}_f$:

$$\begin{aligned} \dot{\hat{\theta}}_{fi} &= -\mu_{fi} s_i \xi_{fi}(x), \\ \dot{\hat{\varepsilon}}_f &= \eta \|s\|, \end{aligned} \quad (49)$$

where $\mu_{fi} > 0$, $i = 1, 2$, and $\eta > 0$.

The properties of the proposed adaptive fuzzy ITSMC law is summarized by the following theorem.

Theorem 11. *For the dynamic model (2) of the NWMR with an unknown function $f(x)$ (26), if the integral terminal sliding mode is chosen as (22), the actuator voltage control input with dynamic robust controller u_r (46) is designed as (45), and the adaptation laws are (49); then*

- (1) all the signals in the closed system are bounded;
- (2) the sliding variable S will converge to the neighborhood of the integral terminal sliding mode $S = 0$ as $\|S\| \leq \delta = \min\{\delta_1, \delta_2\}$ in finite time, where

$$\begin{aligned} \delta_1 &= \frac{\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)\| + \|\tilde{\varepsilon}_f\|}{\lambda_1}, \\ \delta_2 &= \left[\frac{\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)\| + \|\tilde{\varepsilon}_f\|}{\lambda_2} \right]^{1/\rho}. \end{aligned} \quad (50)$$

Moreover, the auxiliary velocity tracking error e_{vi} will converge to the region $|e_{vi}| \leq 2\delta + |e_{vi}(0)|$, $i = 1, 2$ in finite time.

Proof. Consider the following Lyapunov function candidate:

$$V_3 = V_{31} + V_{32}, \quad (51)$$

where

$$\begin{aligned} V_{31} &= \frac{1}{2} S^T \overline{M}(q) S, \\ V_{32} &= \frac{1}{2} \left(\sum_{i=1}^2 \frac{1}{\mu_{fi}} \tilde{\theta}_{fi}^T \tilde{\theta}_{fi} + \frac{1}{\eta} \tilde{\varepsilon}_f^2 \right). \end{aligned} \quad (52)$$

(1) Differentiating V_{31} with respect to time, using (48) and Property 3, we have

$$\begin{aligned} \dot{V}_{31} = & -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho - S^T u_r + \sum_{i=1}^2 s_i \omega_{fi}(x) \\ & + \sum_{i=1}^2 s_i \tilde{\theta}_{fi}^T \xi_{fi}(x). \end{aligned} \quad (53)$$

Note that

$$\sum_{i=1}^2 s_i \omega_{fi}(x) \leq \|S\| \sum_{i=1}^2 \bar{\omega}_i. \quad (54)$$

Substituting (46) and (54) into (53), the following inequality holds:

$$\begin{aligned} \dot{V}_{31} \leq & -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho - \|S\| (\hat{\varepsilon}_f + \sigma) \\ & + \|S\| \sum_{i=1}^2 \bar{\omega}_i + \sum_{i=1}^2 s_i \tilde{\theta}_{fi}^T \xi_{fi}(x) \\ = & -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho - \|S\| \sigma - \|S\| \tilde{\varepsilon}_f \\ & + \sum_{i=1}^2 s_i \tilde{\theta}_{fi}^T \xi_{fi}(x). \end{aligned} \quad (55)$$

There results

$$\begin{aligned} \dot{V}_{31} \leq & -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho - \|S\| \tilde{\varepsilon}_f \\ & + \sum_{i=1}^2 s_i \tilde{\theta}_{fi}^T \xi_{fi}(x). \end{aligned} \quad (56)$$

Differentiating V_{32} with respect to time yields

$$\dot{V}_{32} = \sum_{i=1}^2 \frac{1}{\mu_{fi}} \tilde{\theta}_{fi}^T \dot{\tilde{\theta}}_{fi} + \frac{1}{\eta} \tilde{\varepsilon}_f \dot{\tilde{\varepsilon}}_f. \quad (57)$$

Combining (56) with (57), we can get

$$\begin{aligned} \dot{V}_{31} + \dot{V}_{32} \leq & -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho \\ & + \sum_{i=1}^2 \tilde{\theta}_{fi}^T \left(s_i \xi_{fi}(x) + \frac{1}{\mu_{fi}} \dot{\tilde{\theta}}_{fi} \right) \\ & + \tilde{\varepsilon}_f \left(\frac{1}{\eta} \dot{\tilde{\varepsilon}}_f - \|S\| \right). \end{aligned} \quad (58)$$

That is,

$$\begin{aligned} \dot{V}_{31} + \dot{V}_{32} \leq & -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho \\ & + \sum_{i=1}^2 \tilde{\theta}_{fi}^T \left(s_i \xi_{fi}(x) - \frac{1}{\mu_{fi}} \dot{\tilde{\theta}}_{fi} \right) \\ & + \tilde{\varepsilon}_f \left(\frac{1}{\eta} \dot{\tilde{\varepsilon}}_f - \|S\| \right). \end{aligned} \quad (59)$$

Applying the adaptation laws (49) into (59), one has

$$\dot{V}_{31} + \dot{V}_{32} \leq -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho. \quad (60)$$

Clearly, $\dot{V}_3 = \dot{V}_{31} + \dot{V}_{32} \leq 0$, it is concluded that all the signals s_i , e_{vi} , $\hat{\theta}_{fi}$, and $\hat{\varepsilon}_f$ are bounded.

(2) According to (56), one gets

$$\begin{aligned}\dot{V}_{31} &\leq -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho - \|S\| \tilde{\varepsilon}_f \\ &\quad + S^T [\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)] \\ &= -S^T K_1 S - S^T K_2 \text{sig}(S)^\rho \\ &\quad + S^T \left[\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) - \frac{\tilde{\varepsilon}_f}{\|S\|} S \right],\end{aligned}\quad (61)$$

which can be further changed into the following two forms:

$$\begin{aligned}\dot{V}_{31} &< -S^T \left\{ K_1 - \text{diag} \left[\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) \right. \right. \\ &\quad \left. \left. - \frac{\tilde{\varepsilon}_f}{\|S\|} S \right] \text{diag}^{-1}(S) \right\} S - S^T K_2 \text{sig}(S)^\rho,\end{aligned}\quad (62)$$

$$\begin{aligned}\dot{V}_{31} &< -S^T K_1 S - S^T \left\{ K_2 - \text{diag} \left[\hat{f}(x | \theta_f^*) \right. \right. \\ &\quad \left. \left. - \hat{f}(x | \hat{\theta}_f) - \frac{\tilde{\varepsilon}_f}{\|S\|} S \right] \text{diag}^{-1}[\text{sig}(S)^\rho] \right\} \\ &\quad \cdot \text{sig}(S)^\rho.\end{aligned}\quad (63)$$

Denote

$$\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) - \frac{\tilde{\varepsilon}_f}{\|S\|} S = [\tilde{f}_1, \tilde{f}_2]^T. \quad (64)$$

For (62), if $\lambda_1 > |\tilde{f}_i|/|s_i|$, $i = 1, 2$, which means the matrix $K_1 - \text{diag}[\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) - (\tilde{\varepsilon}_f/\|S\|)S] \text{diag}^{-1}(S)$ is positive definite, the similar structure as (35) is kept. Hence, finite time stability is guaranteed. Otherwise, $|s_i| \leq |\tilde{f}_i|/|\lambda_1|$, $i = 1, 2$. We can conclude that

$$\begin{aligned}\|S\|^2 &= s_1^2 + s_2^2 \leq \frac{\tilde{f}_1^2 + \tilde{f}_2^2}{\lambda_1^2} \\ &= \frac{\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) - (\tilde{\varepsilon}_f/\|S\|)S\|^2}{\lambda_1^2} \\ &\leq \frac{(\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)\| + \|\tilde{\varepsilon}_f\|)^2}{\lambda_1^2},\end{aligned}\quad (65)$$

that is,

$$\|S\| \leq \frac{\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)\| + \|\tilde{\varepsilon}_f\|}{\lambda_1} = \delta_1. \quad (66)$$

Therefore, the region $\|S\| \leq \delta_1$ can be reached in finite time.

For (63), if $\lambda_2 > |\tilde{f}_i|/|s_i|^\rho$, $i = 1, 2$, which means the matrix $K_2 - \text{diag}[\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) - (\tilde{\varepsilon}_f/\|S\|)S] \text{diag}^{-1}(S^\rho)$ is positive definite, the similar structure as (35) is kept.

Hence, finite time stability is guaranteed. Otherwise, $|s_i|^\rho \leq |\tilde{f}_i|/|\lambda_2|$, $i = 1, 2$. We can conclude from Lemma 7 that

$$\begin{aligned}\|S\|^{4\rho} &= (s_1^2 + s_2^2)^{2\rho} \leq (s_1^{2\rho} + s_2^{2\rho})^2 \leq \left(\frac{\tilde{f}_1^2 + \tilde{f}_2^2}{\lambda_2^2} \right)^2 \\ &= \frac{\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f) - (\tilde{\varepsilon}_f/\|S\|)S\|^4}{\lambda_2^4} \\ &\leq \frac{(\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)\| + \|\tilde{\varepsilon}_f\|)^4}{\lambda_2^4};\end{aligned}\quad (67)$$

that is,

$$\|S\| \leq \left[\frac{\|\hat{f}(x | \theta_f^*) - \hat{f}(x | \hat{\theta}_f)\| + \|\tilde{\varepsilon}_f\|}{\lambda_2} \right]^{1/\rho} = \delta_2. \quad (68)$$

Therefore, the region $\|S\| \leq \delta_2$ can be reached in finite time.

By virtue of (66) and (68), the region $\|S\| \leq \delta = \min\{\delta_1, \delta_2\}$ can be reached in finite time.

When $\|S\| \leq \delta$, for $i = 1, 2$, $|s_i| \leq \delta$. The integral terminal sliding mode (22) can be changed into the following form:

$$\begin{aligned}s_i &= e_{vi} - e_{vi}(0) + \beta_i \int_0^t |e_{vi}|^{\gamma_i} \text{sign}(e_{vi}) d\tau = \phi_i, \\ &|\phi_i| \leq \delta,\end{aligned}\quad (69)$$

or the equivalent form

$$\begin{aligned}&e_{vi} - e_{vi}(0) \\ &+ \left[\beta_i - \frac{\phi_i}{\int_0^t |e_{vi}|^{\gamma_i} \text{sign}(e_{vi}) d\tau} \right] \int_0^t |e_{vi}|^{\gamma_i} \text{sign}(e_{vi}) d\tau \\ &= 0.\end{aligned}\quad (70)$$

If $\beta_i > |\phi_i|/\int_0^t |e_{vi}|^{\gamma_i} \text{sign}(e_{vi}) d\tau$, (70) is kept in the form of the integral terminal sliding mode. Hence, finite time convergence is guaranteed. Otherwise, $|\int_0^t |e_{vi}|^{\gamma_i} \text{sign}(e_{vi}) d\tau| \leq |\phi_i|/\beta_i \leq \delta/\beta_i$; the region

$$\begin{aligned}|e_{vi}| &\leq \beta_i \left| \int_0^t |e_{vi}|^{\gamma_i} \text{sign}(e_{vi}) d\tau \right| + \delta + |e_{vi}(0)| \\ &\leq 2\delta + |e_{vi}(0)|,\end{aligned}\quad (71)$$

can be reached in finite time. \square

Remark 12. Both the control law (27) and the adaptive fuzzy control law (45) contain a nonlinear term $K_1 S + K_2 \text{sig}(S)^\rho$ with the form of fast terminal sliding mode, which assures the boundedness of the signals in the closed system and the finite time convergence of the auxiliary velocity tracking error.

Remark 13. According to (66) and (68), the parameters λ_1 and λ_2 can be chosen large enough to make the boundary δ small. However, increasing the parameters λ_1 and λ_2 will increase the level of control input and will cause implementation problem.

Remark 14. The relationship between the auxiliary velocity tracking error e_{v_i} and the width of the boundary layer δ surrounding the integral terminal sliding mode surface $S = 0$ is given by (69) and (71).

4. Simulation Results

In this section, a simulation will be provided to show the effectiveness of the proposed control strategy.

Referring to [18], the parameters of the NWMR and its actuators are chosen as $m = 10$ kg, $I = 5$ kg·m², $R = 0.12$ m, $r = 0.067$ m, $d = 0.3$ m, $N = 50$, $K_b = 0.026$, $K_T = 0.026$ Nm/A, and $R_a = 3.5$ Ω . The surface friction and the external disturbance are generated by $\bar{F}(\vartheta) + \bar{\tau}_d = [5 \sin(2t), 5 \sin(2t)]^T$. In this simulation, the initial posture and velocity of the practical NWMR are taken as $q(0) = [0.1, 0.2, 0]^T$ and $\vartheta = [\nu, \omega]^T = [0, 0]^T$, respectively.

The reference linear velocity and angular velocity are defined as $\nu_r(t) = 1$ and $\omega_r(t) = 1$. The trajectory of the reference NWMR is defined as

$$\begin{aligned}\dot{x}_r &= \nu_r \cos \theta_r - \omega_r d \sin \theta_r, \\ \dot{y}_r &= \nu_r \sin \theta_r + \omega_r d \cos \theta_r, \\ \dot{\theta}_r &= \omega_r.\end{aligned}\quad (72)$$

The initial posture of the reference NWMR is taken as $q_r(0) = [0, 0, 0]^T$.

The objective of the trajectory tracking control is to design a strategy such that $q(t)$ converges asymptotically to $q_r(t)$, while all signals in the derived closed-loop system are able to remain bounded. In the proposed control strategy, an auxiliary velocity controller ϑ_c is designed for the kinematic model to meet the control objective. Then, the actuator voltage control input u is designed for the dynamic model such that ϑ converges to ϑ_c which is designed at the first step in finite time.

In the actuator voltage control input u , the nonlinear function

$$\begin{aligned}f(x) &= [f_1(x), f_2(x)]^T \\ &= \bar{M}(q) \dot{\vartheta}_c + \bar{M}(q) \Lambda \text{sig}(e_\vartheta)^\gamma + \bar{V}(q, \dot{q}) \vartheta_c \\ &\quad - \bar{V}(q, \dot{q}) e_\vartheta(0) + \bar{V}(q, \dot{q}) \Lambda \int_0^t \text{sig}(e_\vartheta)^\gamma d\tau \\ &\quad + \bar{F}(\vartheta) + \bar{\tau}_d\end{aligned}\quad (73)$$

is contained, where $x = [\vartheta_c^T, \dot{\vartheta}_c^T, \vartheta^T]^T$.

We suppose that there is no prior information of the robot parameters such as mass, moment of inertial, friction coefficients, and the external disturbance; that is, the nonlinear function $f_i(x)$, $i = 1, 2$, is assumed to be completely unknown. Two fuzzy systems in the form of (9) are used to approximate $f_1(x)$ and $f_2(x)$. The fuzzy systems have $x_1 = \nu_c$, $x_2 = \omega_c$, $x_3 = \dot{\nu}_c$, $x_4 = \dot{\omega}_c$, $x_5 = \nu$, and $x_6 = \omega$ as inputs;

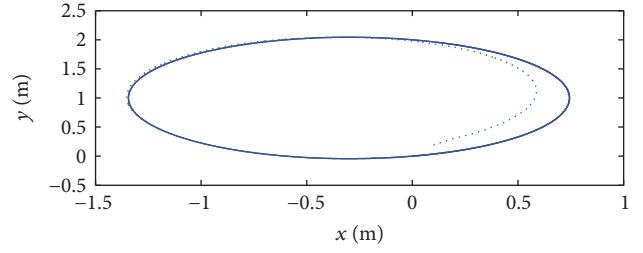


FIGURE 2: Reference trajectory (—) and actual trajectory (···).

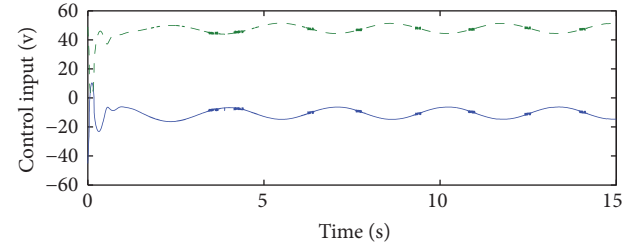


FIGURE 3: Actuator voltages u_1 (—) and u_2 (---).

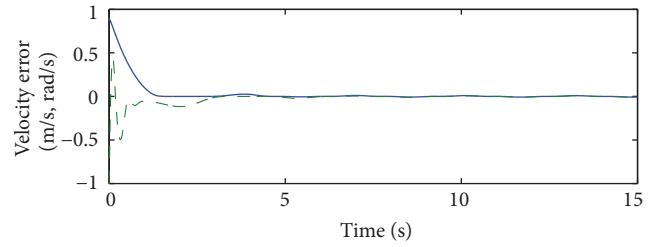


FIGURE 4: Auxiliary velocity tracking errors e_{ϑ_1} (—) and e_{ϑ_2} (---).

the fuzzy membership functions for each variable x_i , $i = 1, 2, \dots, 6$, are chosen as

$$\begin{aligned}\mu_{F_1^1}(x_i) &= \exp \left[-\frac{1}{2} \left(\frac{x_i + 1.25}{0.6} \right)^2 \right], \\ \mu_{F_1^2}(x_i) &= \exp \left[-\frac{1}{2} \left(\frac{x_i}{0.6} \right)^2 \right], \\ \mu_{F_1^3}(x_i) &= \exp \left[-\frac{1}{2} \left(\frac{x_i - 1.25}{0.6} \right)^2 \right].\end{aligned}\quad (74)$$

The initial values of the estimated parameters $\hat{\theta}_{fi}(0)$, $i = 1, 2$, and $\hat{\varepsilon}_f(0)$ are all set to 0.01.

Referring to [1, 31, 36], the parameters of the control law are chosen as $k_1 = 1$, $k_2 = 20$, $k_3 = 10$, $\beta_1 = \beta_2 = 1$, $\gamma_1 = \gamma_2 = 0.5$, $k_{11} = k_{12} = k_{21} = k_{22} = 20$, $\rho = 0.3$, $\mu_{f1} = \mu_{f2} = 0.5$, $\eta = 0.001$, and $\sigma = 0.1$.

Using our control strategy to control the NWMR, the simulation results are shown in Figures 2–5. Figure 2 is the trajectory tracking process in X-Y plane of the NWMR, Figure 3 is the actuator voltage control input, Figure 4 is the auxiliary velocity tracking errors, and Figure 5 is the tracking position errors, respectively. From Figure 4, it can be observed that the auxiliary velocity tracking error e_{ϑ_1}

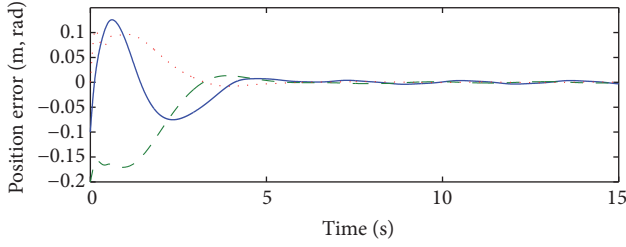
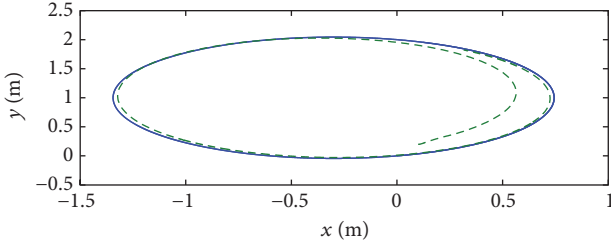
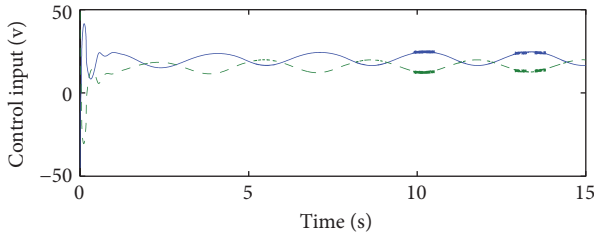
FIGURE 5: Tracking position errors e_1 (-), e_2 (- -), and e_3 (···).

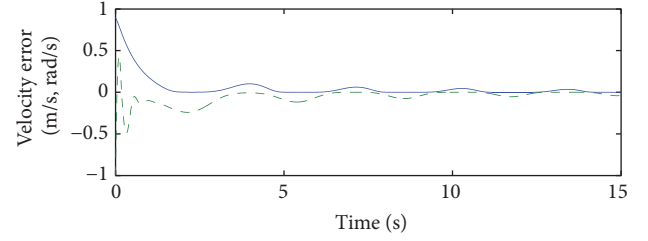
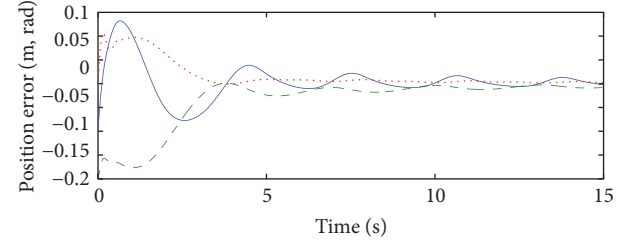
FIGURE 6: Reference trajectory (-) and actual trajectory (- -).

FIGURE 7: Actuator voltages u_1 (-) and u_2 (- -).

converges to $|e_{\theta_1}| \leq 5 \times 10^{-3}$ in finite time $t = 1.389$ s and the auxiliary velocity tracking error e_{θ_2} converges to $|e_{\theta_2}| \leq 7 \times 10^{-3}$ in finite time $t = 3.052$ s, respectively.

In order to compare the proposed integral terminal sliding mode with the integral sliding mode adopted in [25], we use integral sliding mode instead of the integral terminal sliding mode in our control strategy while other design parameters are the same as the corresponding design parameters used in the above simulation. The corresponding simulation results are shown in Figures 6–9. From Figure 8, it can be observed that the auxiliary velocity tracking error e_{θ_1} converges to $|e_{\theta_1}| \leq 3 \times 10^{-2}$ in finite time $t = 13.47$ s and the auxiliary velocity tracking error e_{θ_2} converges to $|e_{\theta_2}| \leq 4 \times 10^{-2}$ in finite time $t = 14.05$ s, respectively.

It is observed that the actual velocity can track the auxiliary velocity in less time using the control strategy proposed in this paper from Figures 4 and 8. As a result, the practical NWMR can track the reference NWMR asymptotically with faster response from Figures 2 and 6. Meanwhile, this favorable performance was obtained with no prior information of the robot parameters such as mass, moment of inertial, friction coefficients, and the external disturbance.

FIGURE 8: Auxiliary velocity tracking errors e_{θ_1} (-) and e_{θ_2} (- -).FIGURE 9: Tracking position errors e_1 (-), e_2 (- -), and e_3 (···).

5. Conclusions

In this paper, a control strategy has been proposed for the trajectory tracking problem of the NWMR with parameter uncertainties and external disturbances. In this study, we take the wheel actuator dynamics into system dynamics and choose the actuator voltage as the control input. The FLS is adopted to estimate the unknown function coming from parameter uncertainties and external disturbances. An adaptive fuzzy integral terminal sliding mode controller is integrated with an auxiliary velocity controller. It has been shown that all the signals in the closed system are bounded and the auxiliary velocity tracking errors converge to a small neighborhood of the origin in finite time. Hence, the tracking position errors converge asymptotically to zeros with faster response than other existing controllers. Simulation results have been provided to show the feasibility of the proposed control strategy. However, wheel skidding and slipping are unavoidable due to tire deformation and other reasons in real environments. In the future, we will extend our results to the trajectory tracking control of the WMR with wheel skidding and slipping.

Conflicts of Interest

The authors declare that they have no conflicts of interest.

Acknowledgments

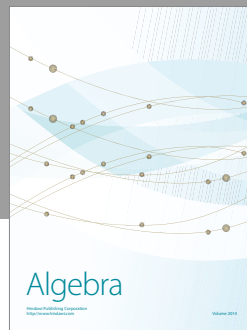
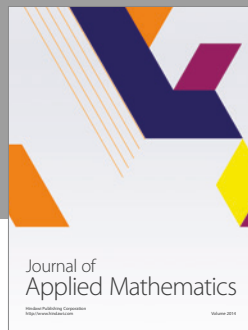
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